



US00RE49313E

(19) **United States**
(12) **Reissued Patent**
Dyas

(10) **Patent Number:** **US RE49,313 E**
(45) **Date of Reissued Patent:** **Nov. 29, 2022**

(54) **FIBROUS PLANT STALK DECORTICATOR**

(71) Applicant: **Bastlab, LLC**, Omaha, NE (US)

(72) Inventor: **Michael R. Dyas**, Waterloo, NE (US)

(73) Assignee: **Bastlab, LLC**, Omaha, NE (US)

(21) Appl. No.: **16/857,387**

(22) Filed: **Apr. 24, 2020**

Related U.S. Patent Documents

Reissue of:

(64) Patent No.: **9,957,642**
Issued: **May 1, 2018**
Appl. No.: **14/733,676**
Filed: **Jun. 8, 2015**

U.S. Applications:

(63) Continuation of application No. 13/807,047, filed as application No. PCT/US2011/042245 on Jun. 28, 2011, now Pat. No. 9,080,257.

(60) Provisional application No. 61/359,089, filed on Jun. 28, 2010.

(51) **Int. Cl.**
D01B 1/30 (2006.01)
D01B 1/28 (2006.01)

(52) **U.S. Cl.**
CPC **D01B 1/28** (2013.01); **D01B 1/30** (2013.01)

(58) **Field of Classification Search**
CPC ... D01B 1/14; D01B 1/28; D01B 1/30; D01B 1/32
See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

398,058 A * 2/1889 Langhammer D01B 1/10
19/29

1,200,615 A * 10/1916 Hubler D01B 1/10
19/12
1,268,975 A * 6/1918 Hubler D01B 1/10
19/12
1,299,946 A * 4/1919 Hubler D01B 1/10
19/12
1,964,633 A 6/1934 Edward
2,162,780 A * 6/1939 Ludington D01B 1/30
19/12
2,197,683 A 4/1940 Burkardt
2,288,652 A * 7/1942 Simons D01B 1/10
19/11
2,341,290 A * 2/1944 Reeves D01B 1/14
19/13
2,402,634 A 6/1946 Karl
2,404,457 A * 7/1946 Reeves D01B 1/14
19/16
2,460,448 A * 2/1949 Cook D01B 1/28
19/30
2,576,406 A * 11/1951 McCrae D01B 1/14
19/12

(Continued)

FOREIGN PATENT DOCUMENTS

FR 1216725 A * 4/1960 D01B 1/28

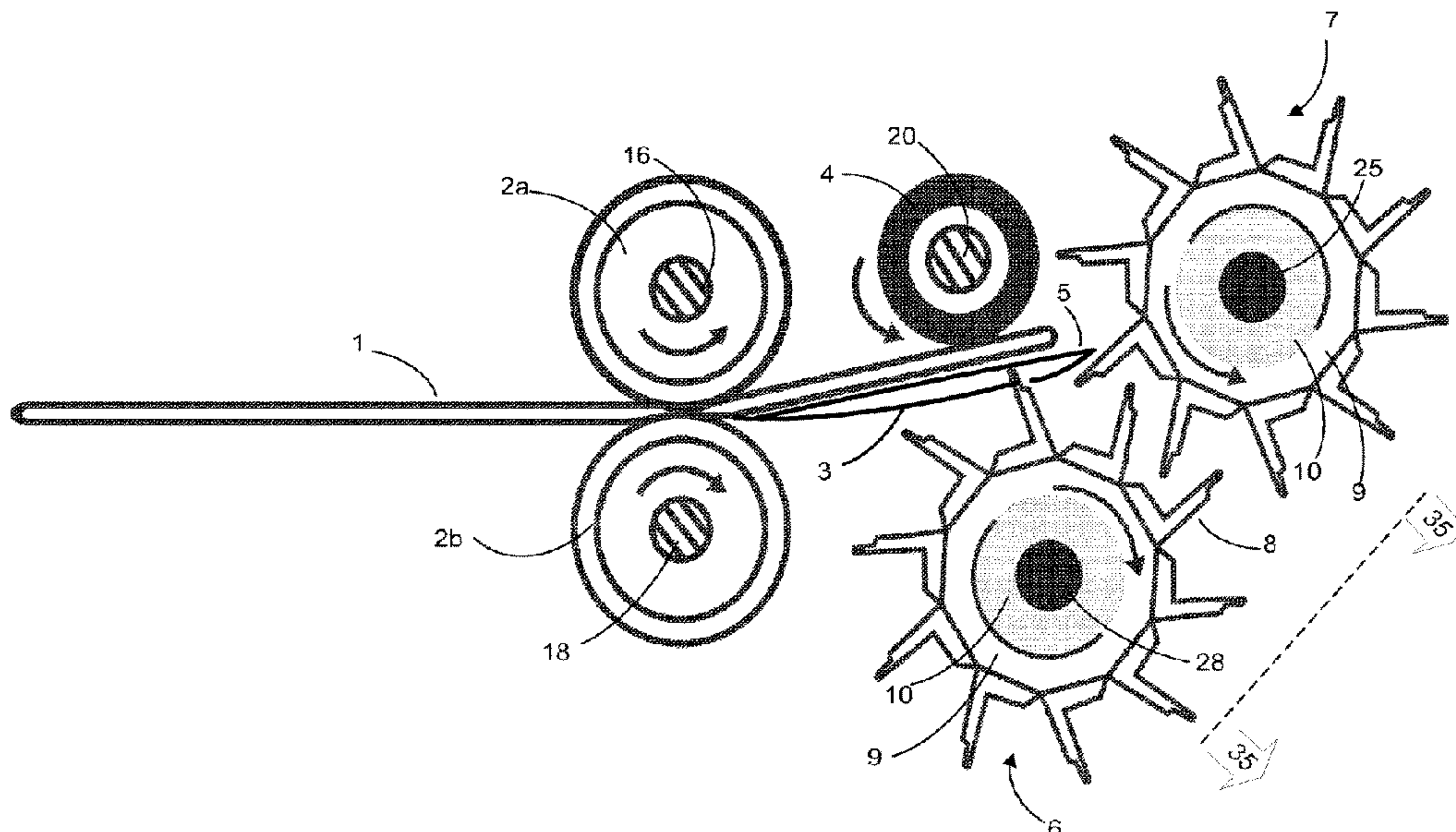
Primary Examiner — Joseph A Kaufman

(74) *Attorney, Agent, or Firm* — Milligan PC LLO

(57) **ABSTRACT**

A fibrous pant stalk decorticator is disclosed. The decorticator includes a feeding device a breaker roll having a first plurality of blades, and a skutcher roll having a second plurality of blades. The skutcher is positioned such that the first and second pluralities of blades overlap. A plurality of air outlets pass through the blades and are configured to provided pressurized air to the fibrous plant stalk as it is passed through the decorticator.

12 Claims, 27 Drawing Sheets



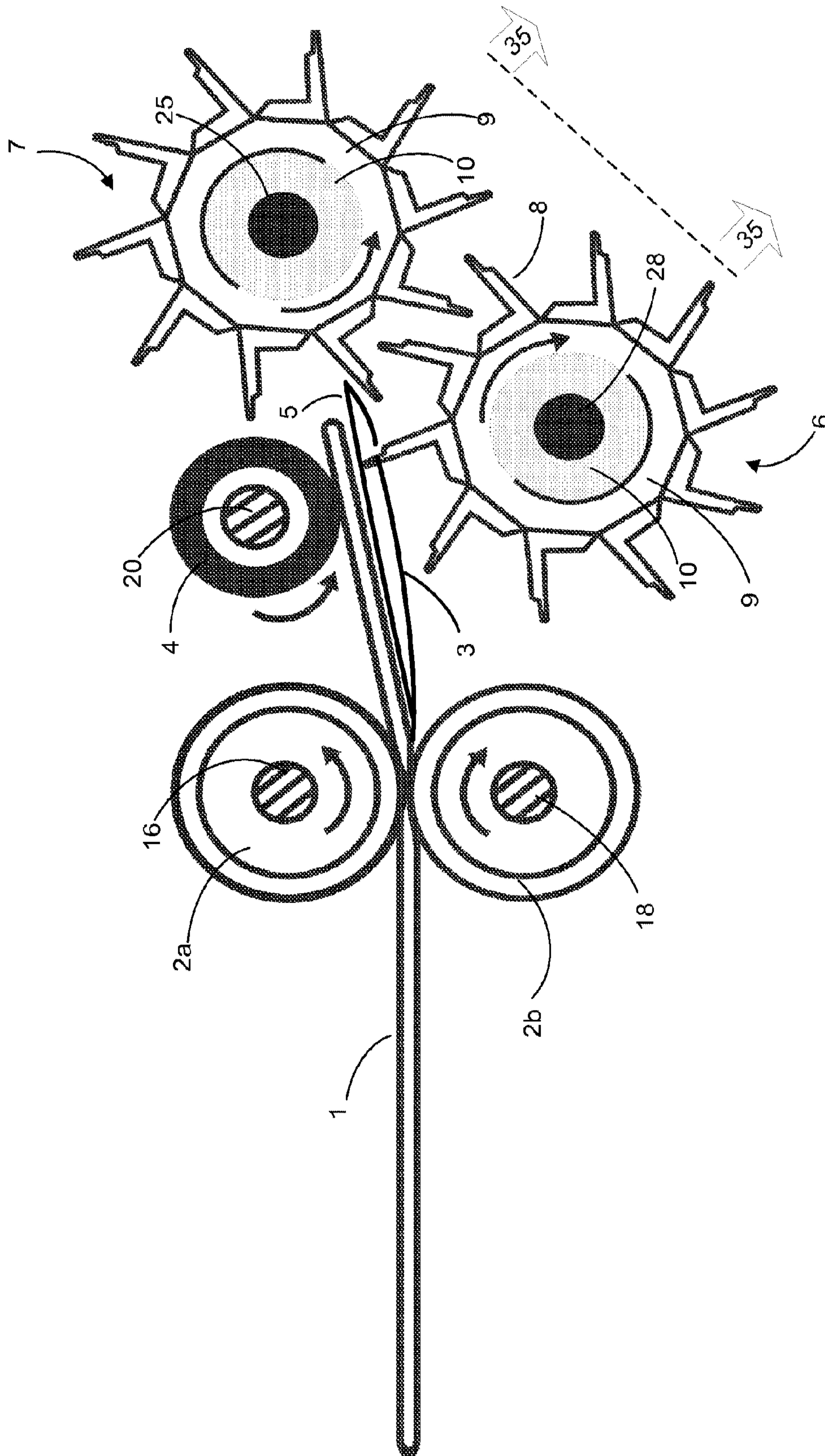
(56)

References Cited

U.S. PATENT DOCUMENTS

2,688,161	A *	9/1954	Michel	D01B 1/28 19/6
3,872,545	A	3/1975	Altosaar	
4,241,474	A	12/1980	Doutre	
5,447,238	A *	9/1995	Aldridge	D01B 1/14 209/254
5,447,276	A *	9/1995	Aldridge	D01B 1/14 241/159
5,507,073	A *	4/1996	Aldridge	D01B 1/14 19/5 A
5,906,030	A *	5/1999	Leduc	D01B 1/22 19/24
6,079,647	A *	6/2000	Leduc	D01B 1/22 19/24
6,357,083	B1 *	3/2002	Stratton	D01B 1/22 19/29
7,152,610	B2 *	12/2006	Csavas	A45D 2/002 132/271
9,080,257	B2 *	7/2015	Lupien	D01B 1/30
2013/0276998	A1 *	10/2013	Park	D21H 11/12 162/20

* cited by examiner



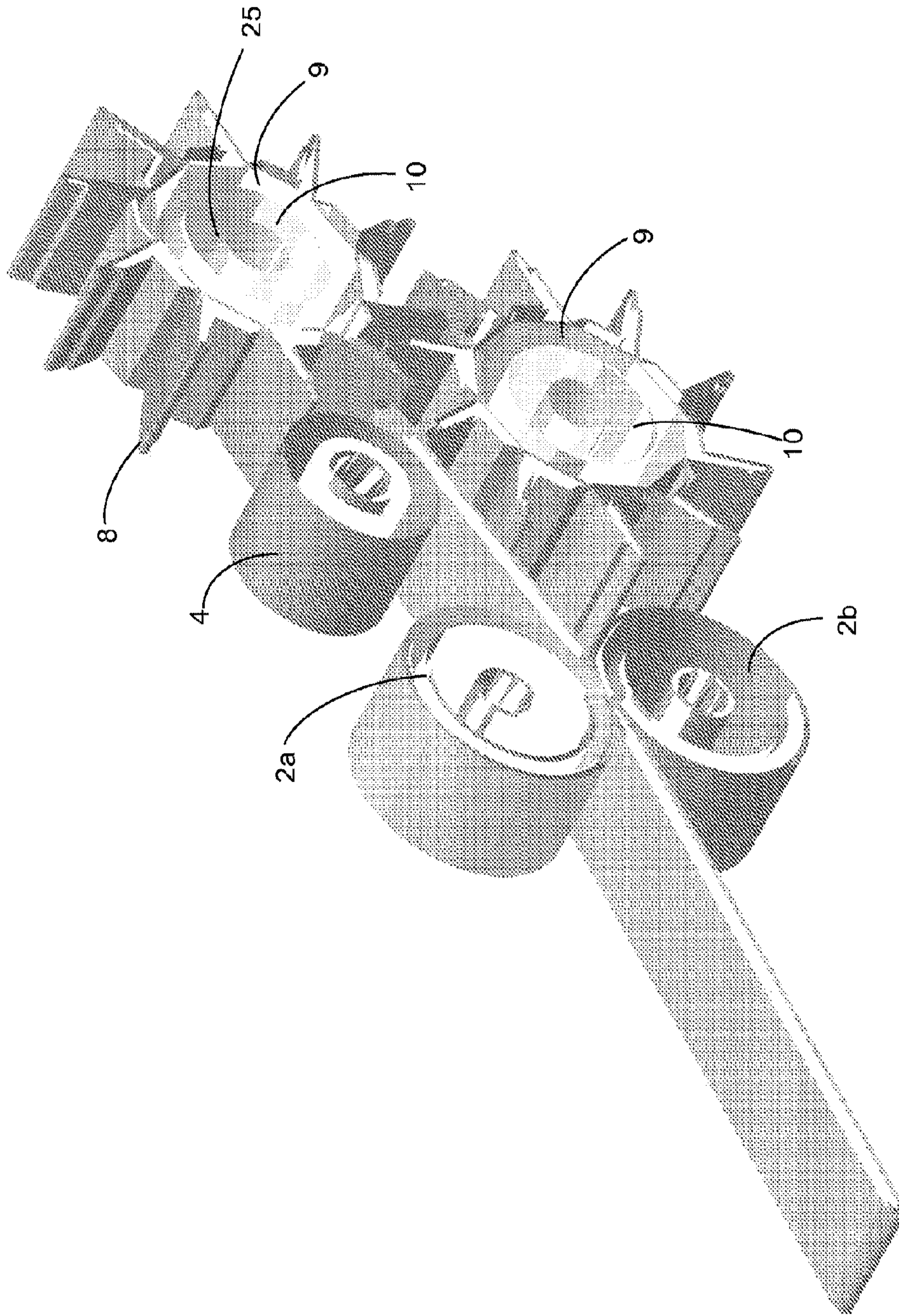


FIG. 2

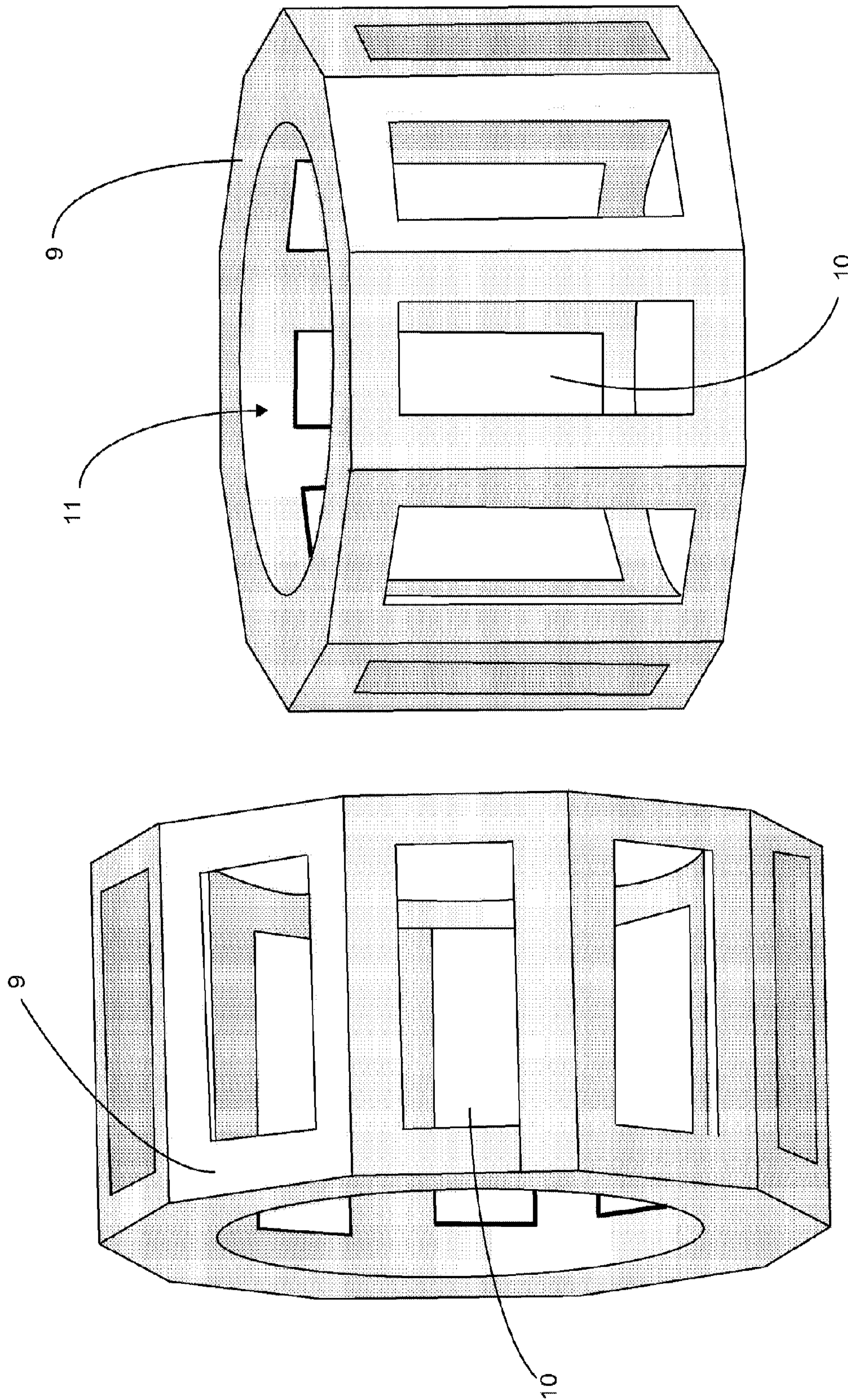


FIG. 3

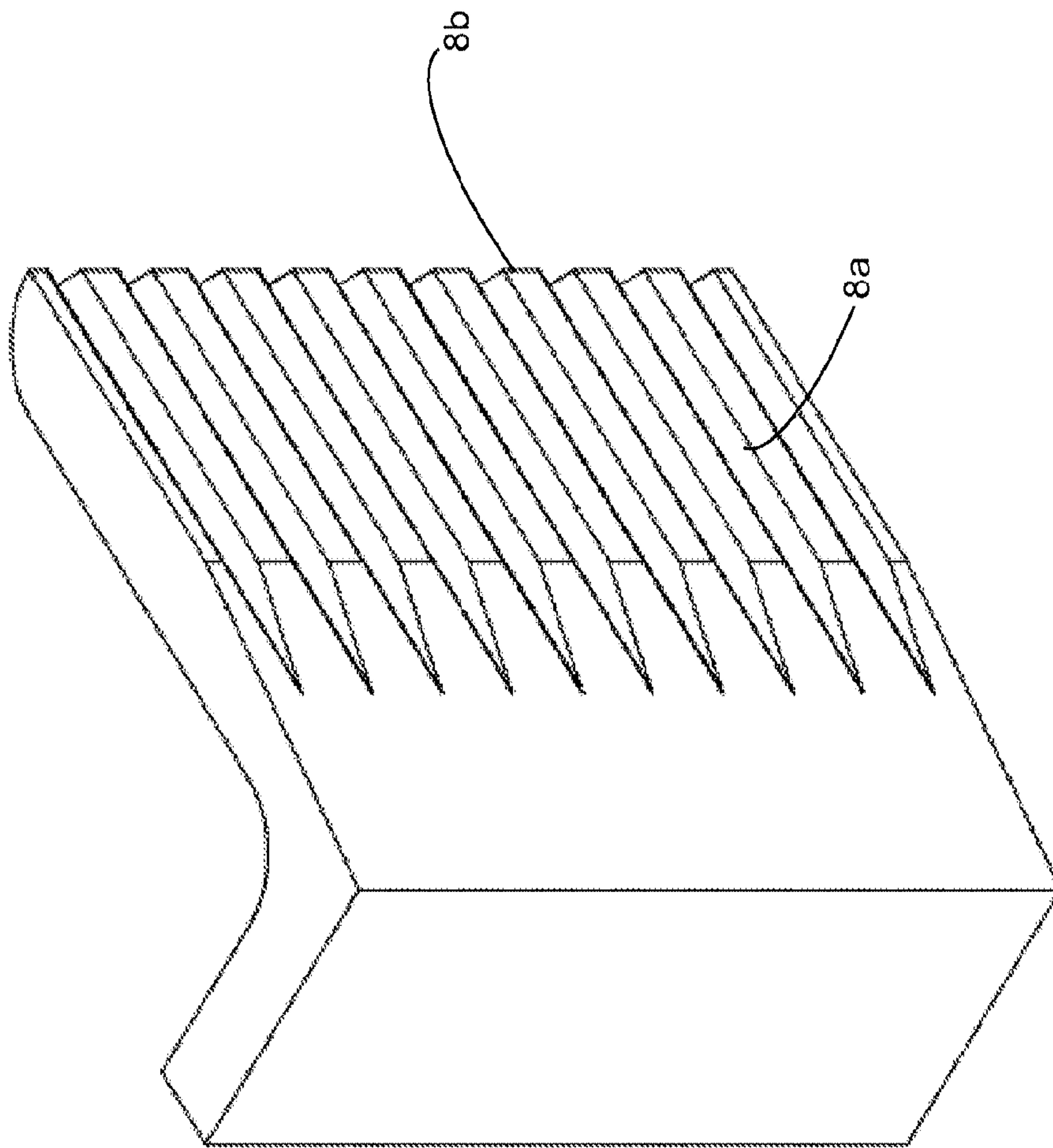


FIG. 4

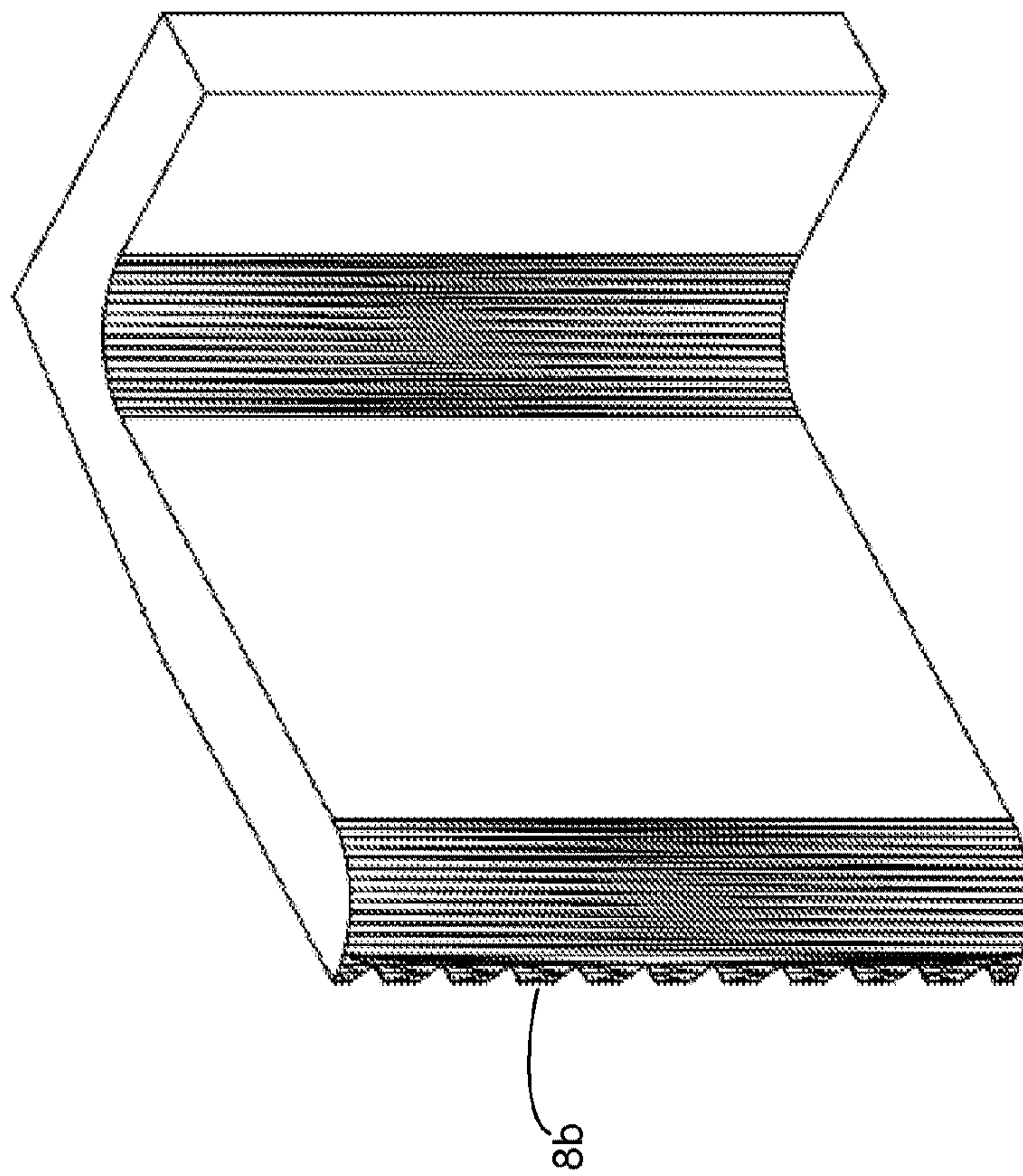


FIG. 5

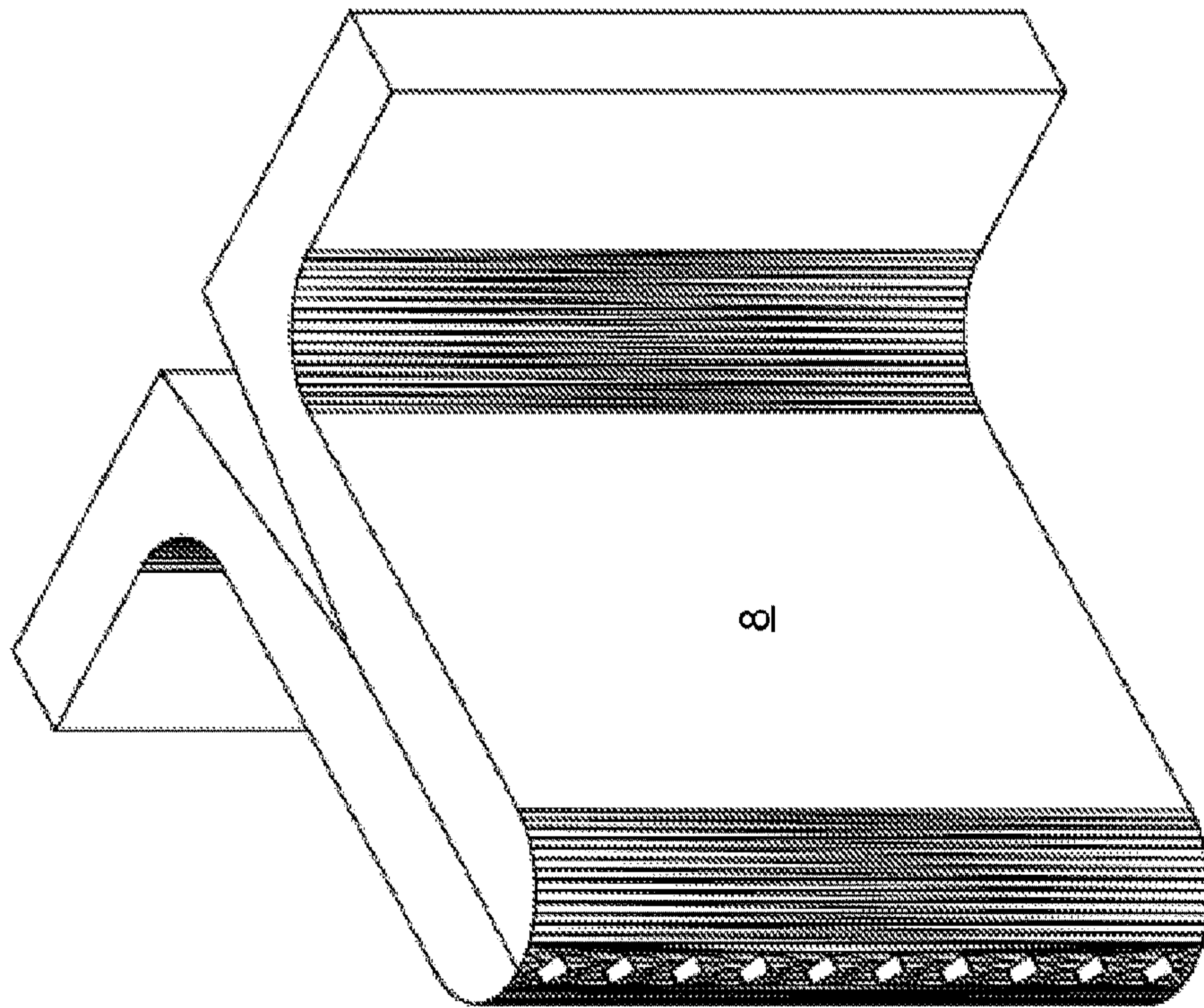


FIG. 6

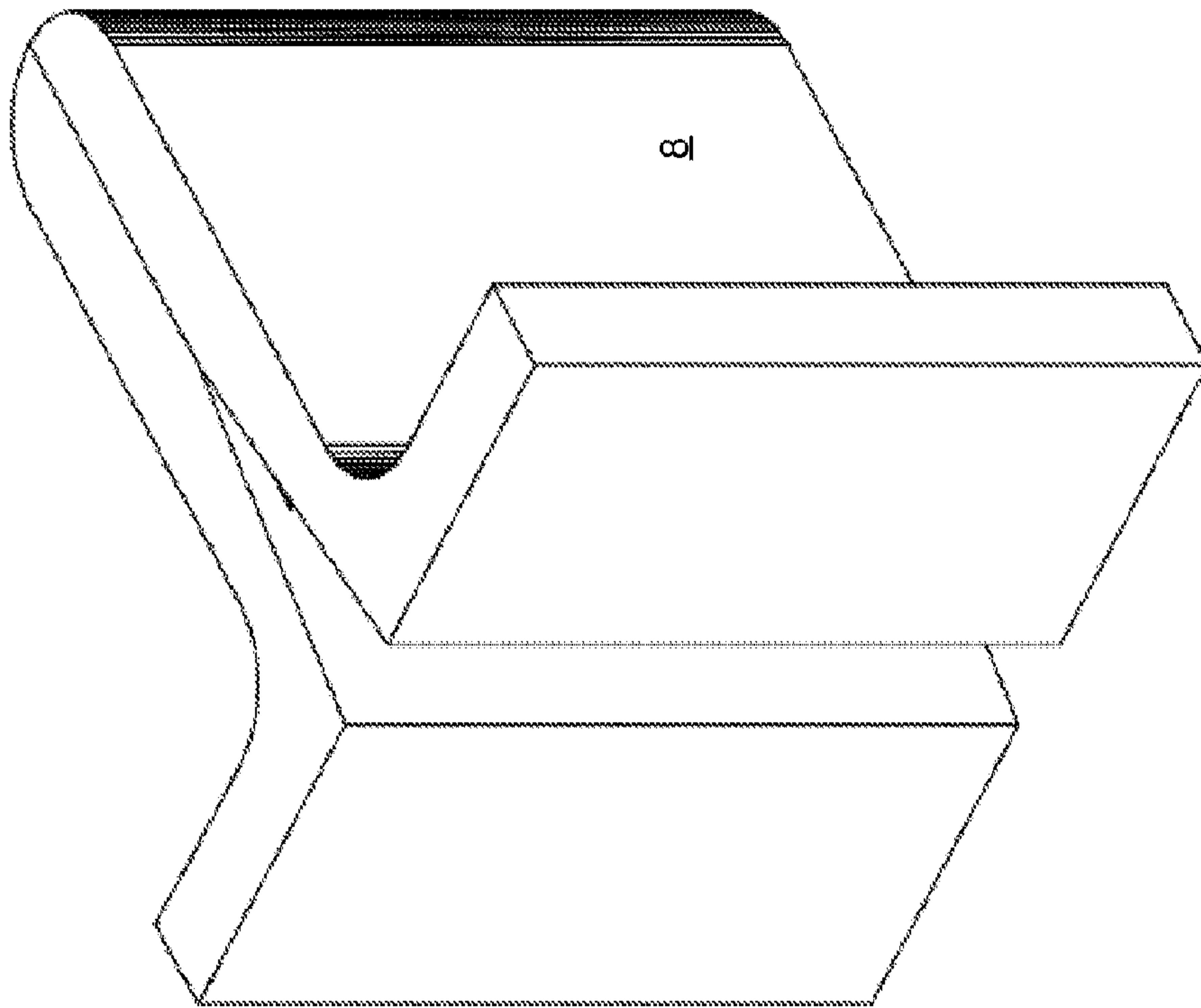


FIG. 7

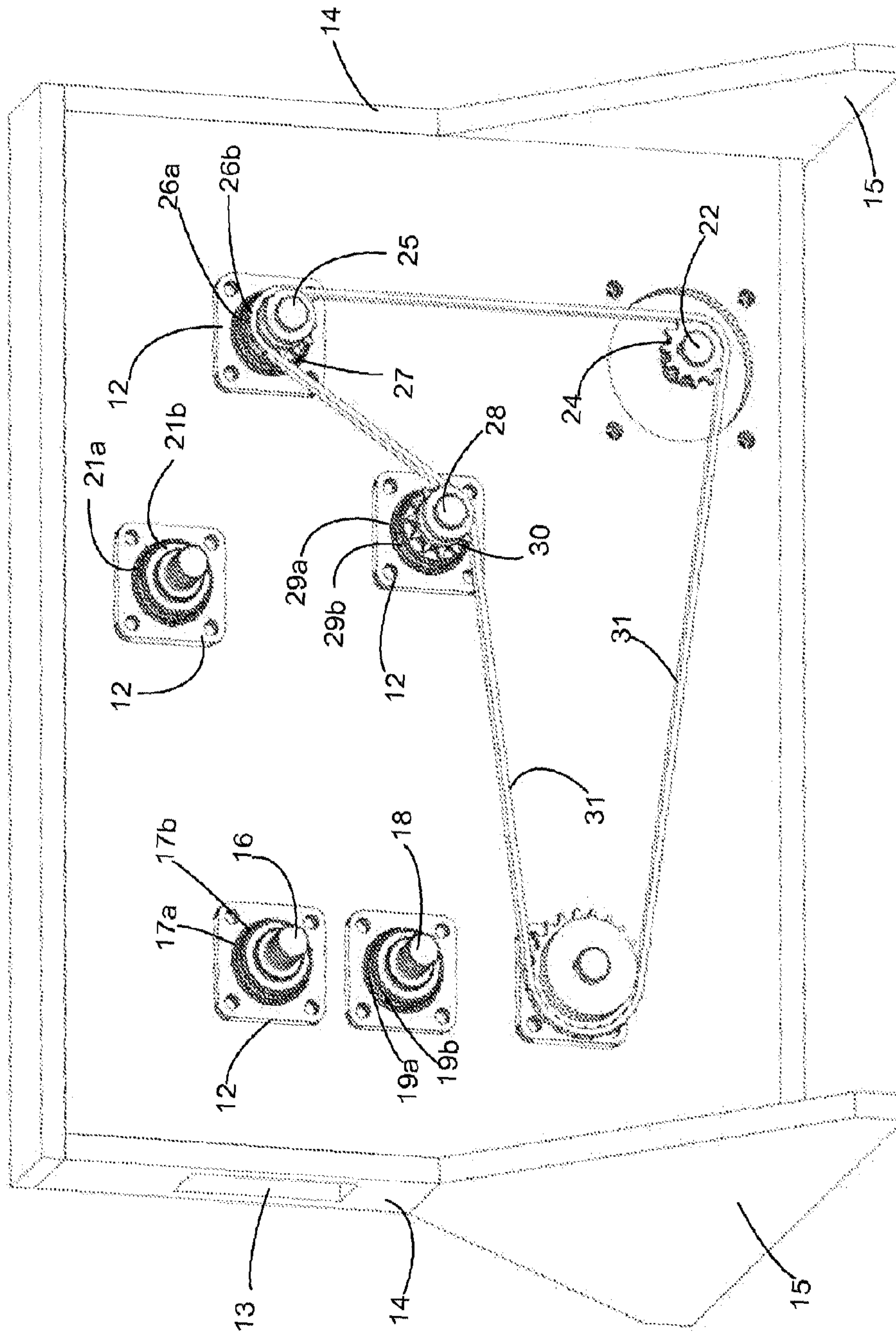


FIG. 8

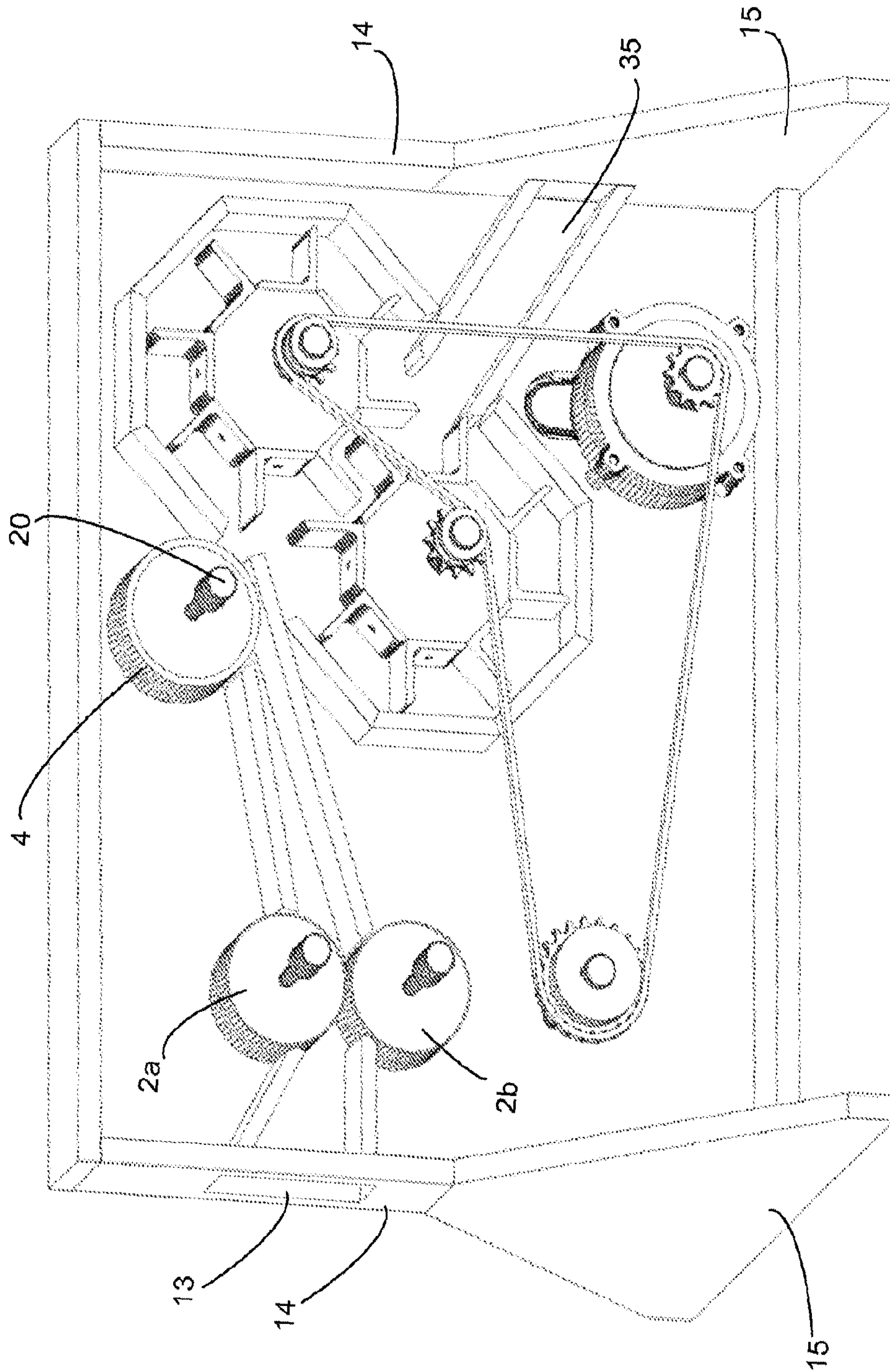


FIG. 9

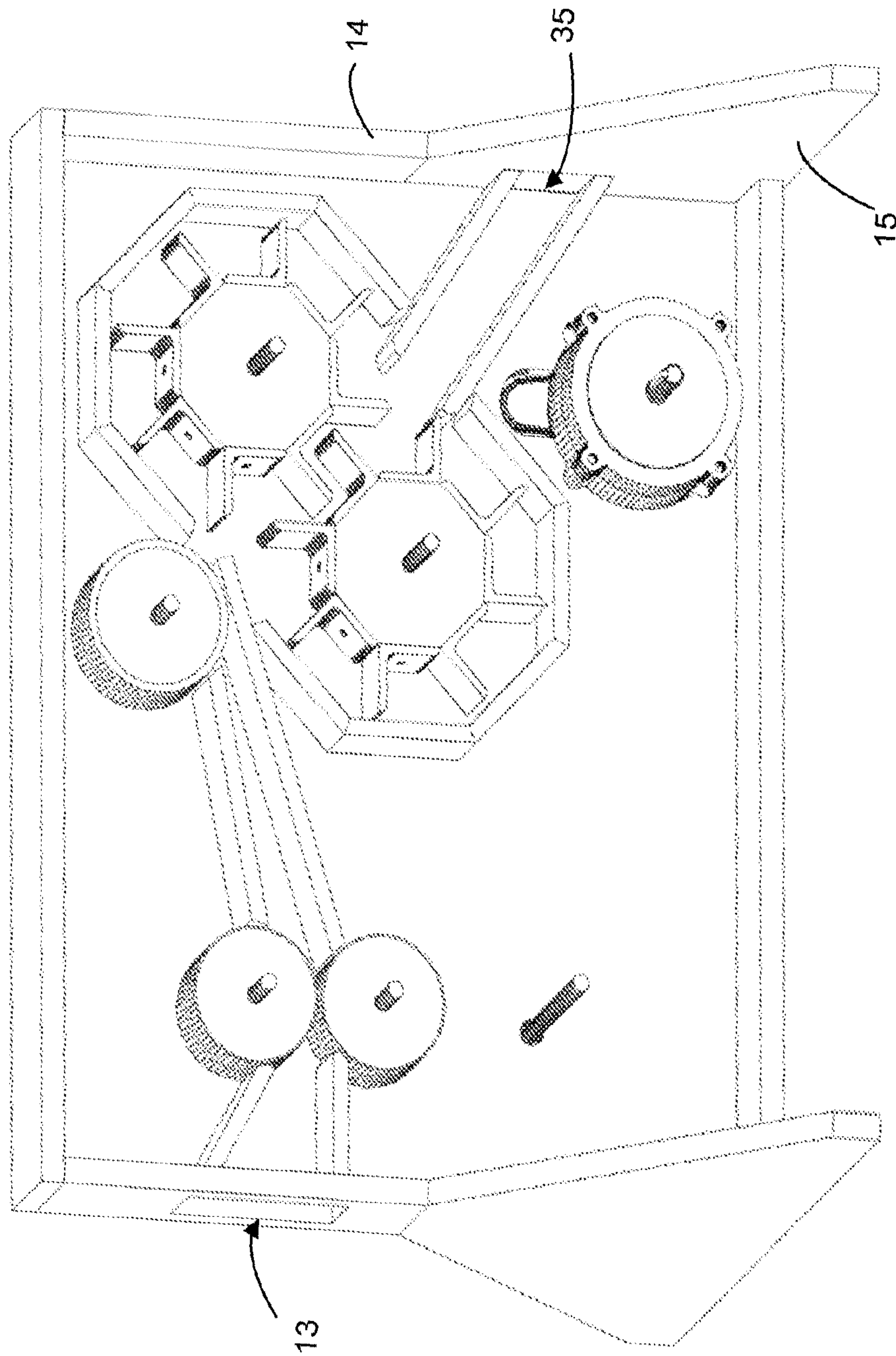


FIG. 10

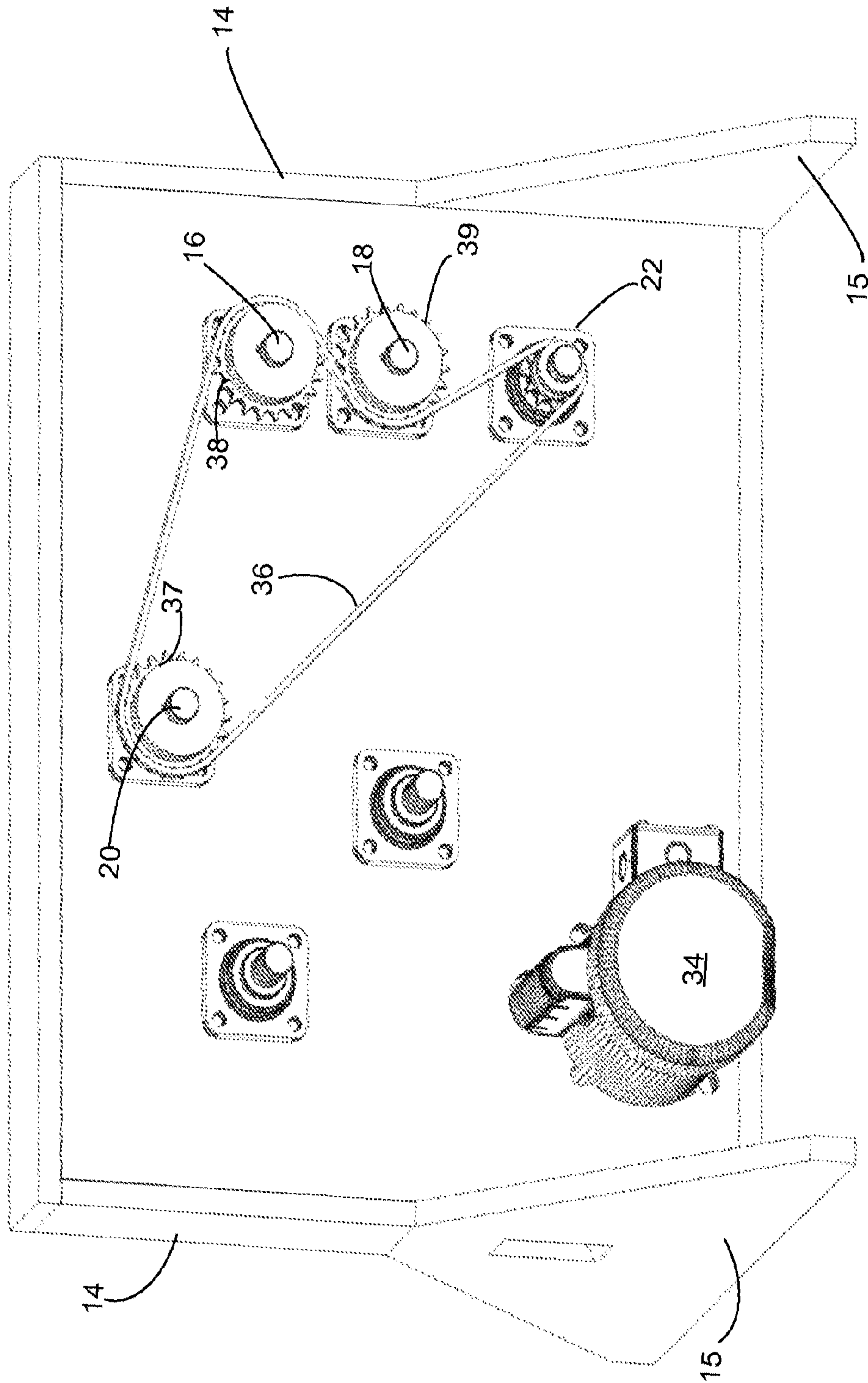


FIG. 11

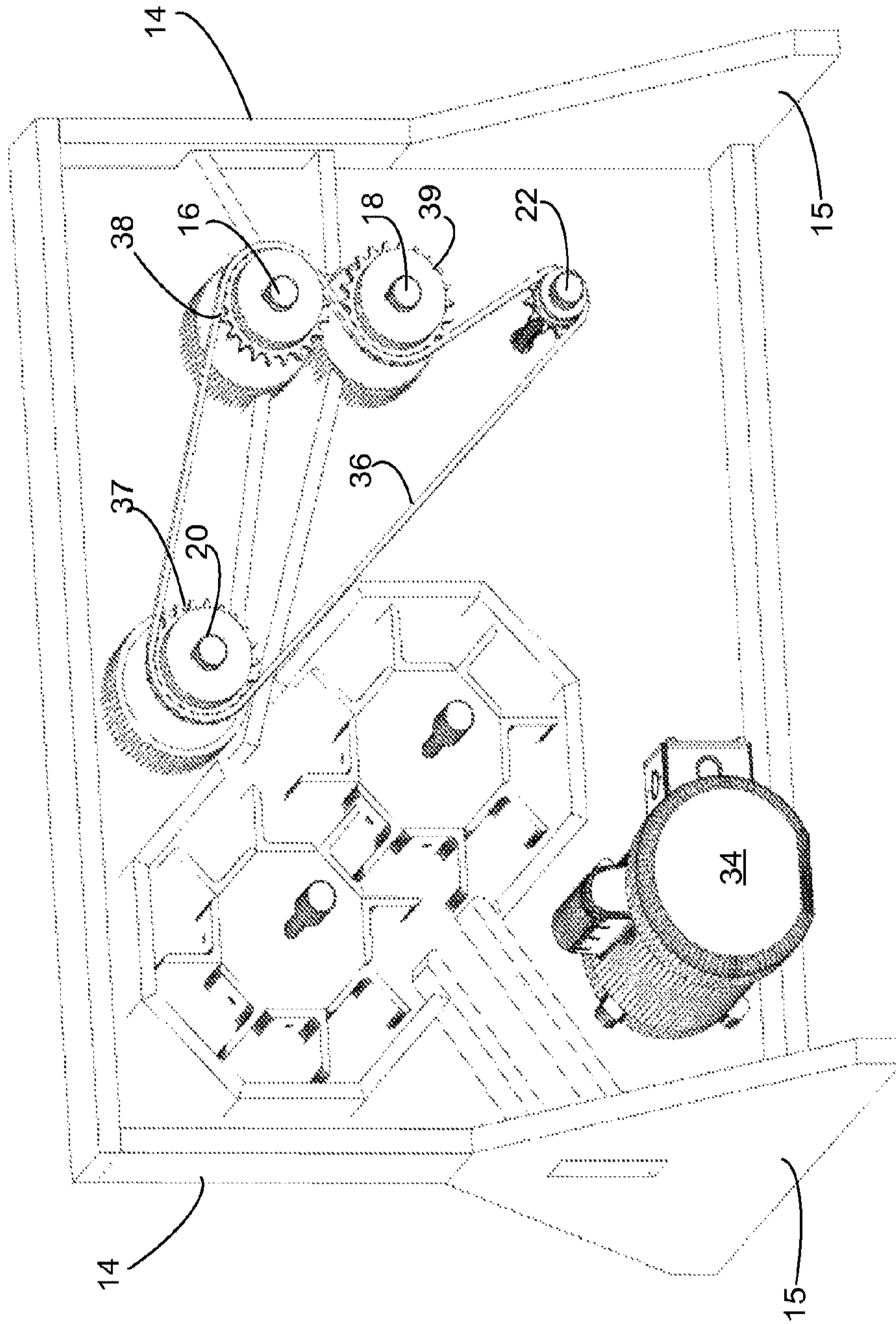


FIG. 12

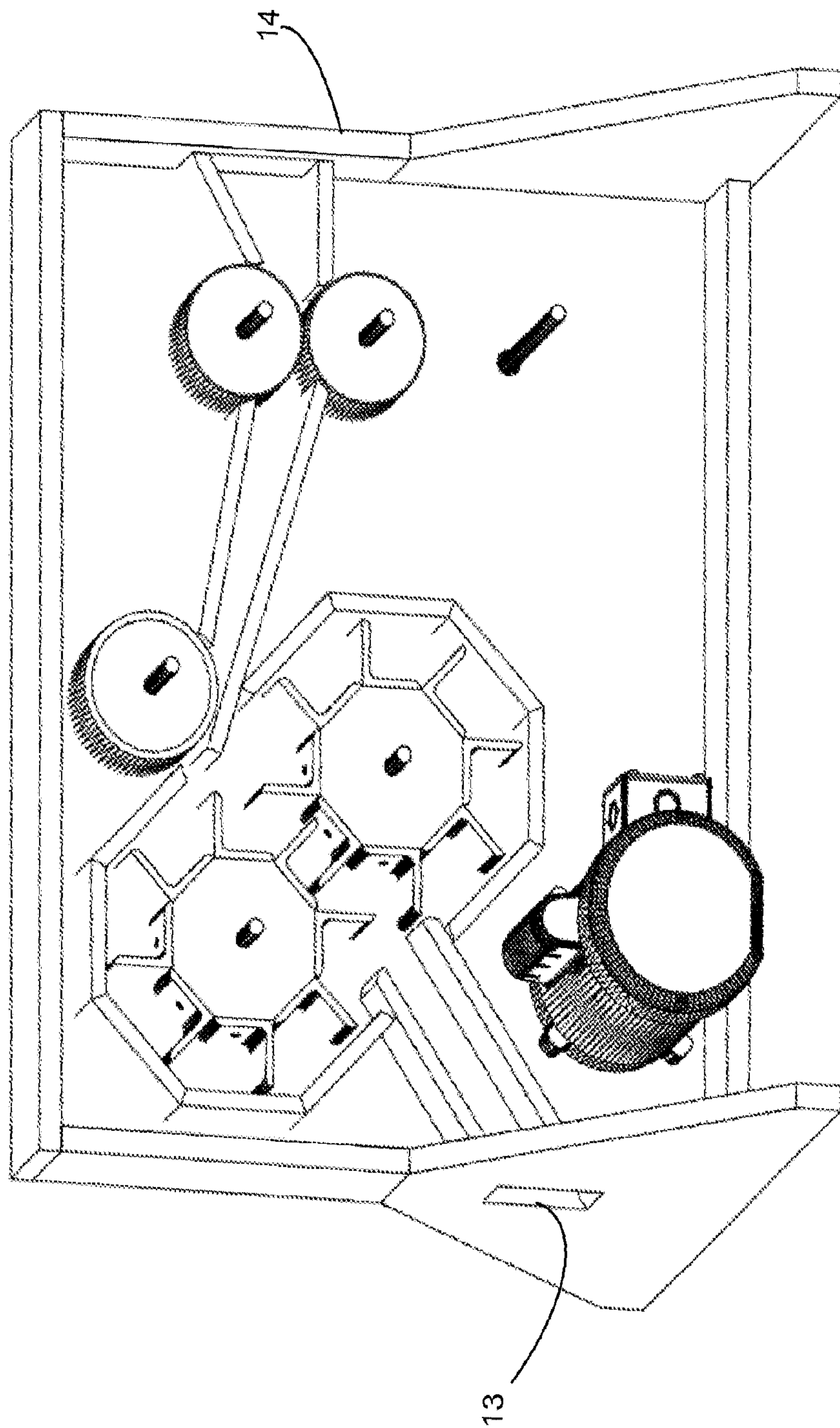


FIG. 13

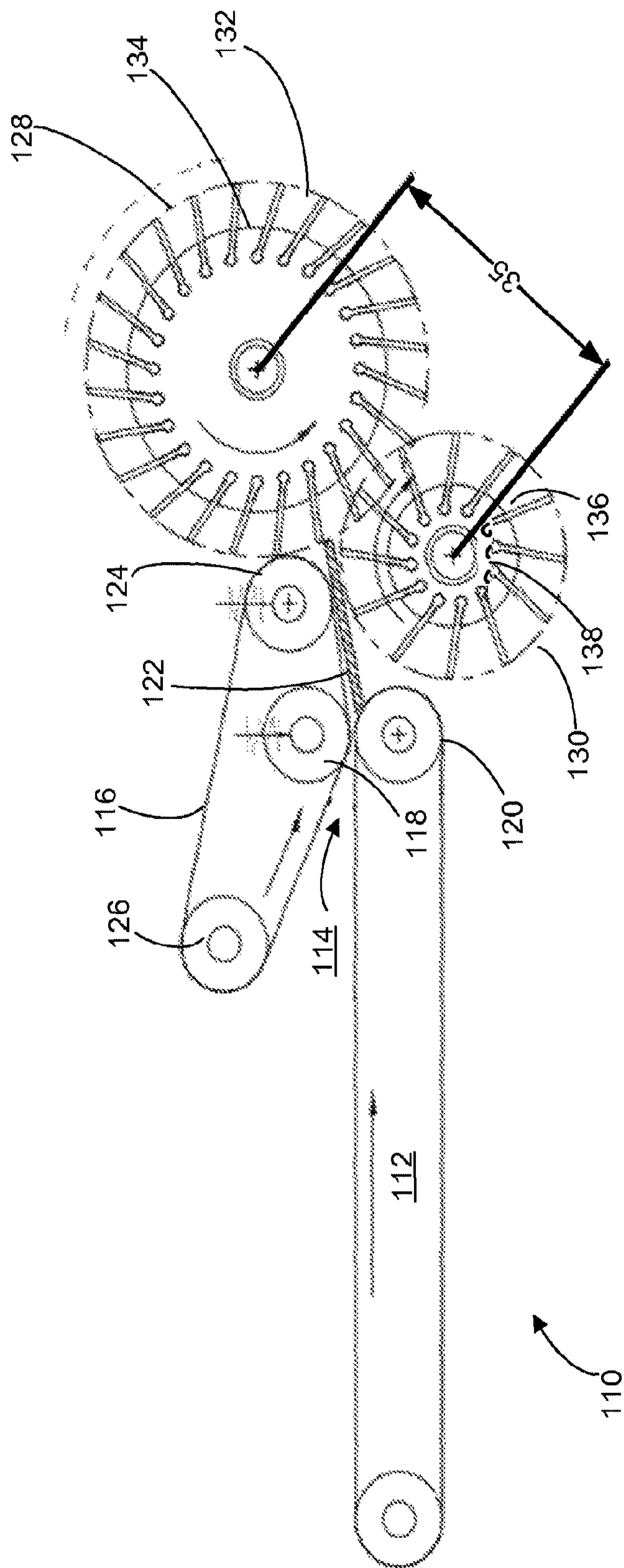


FIG. 14

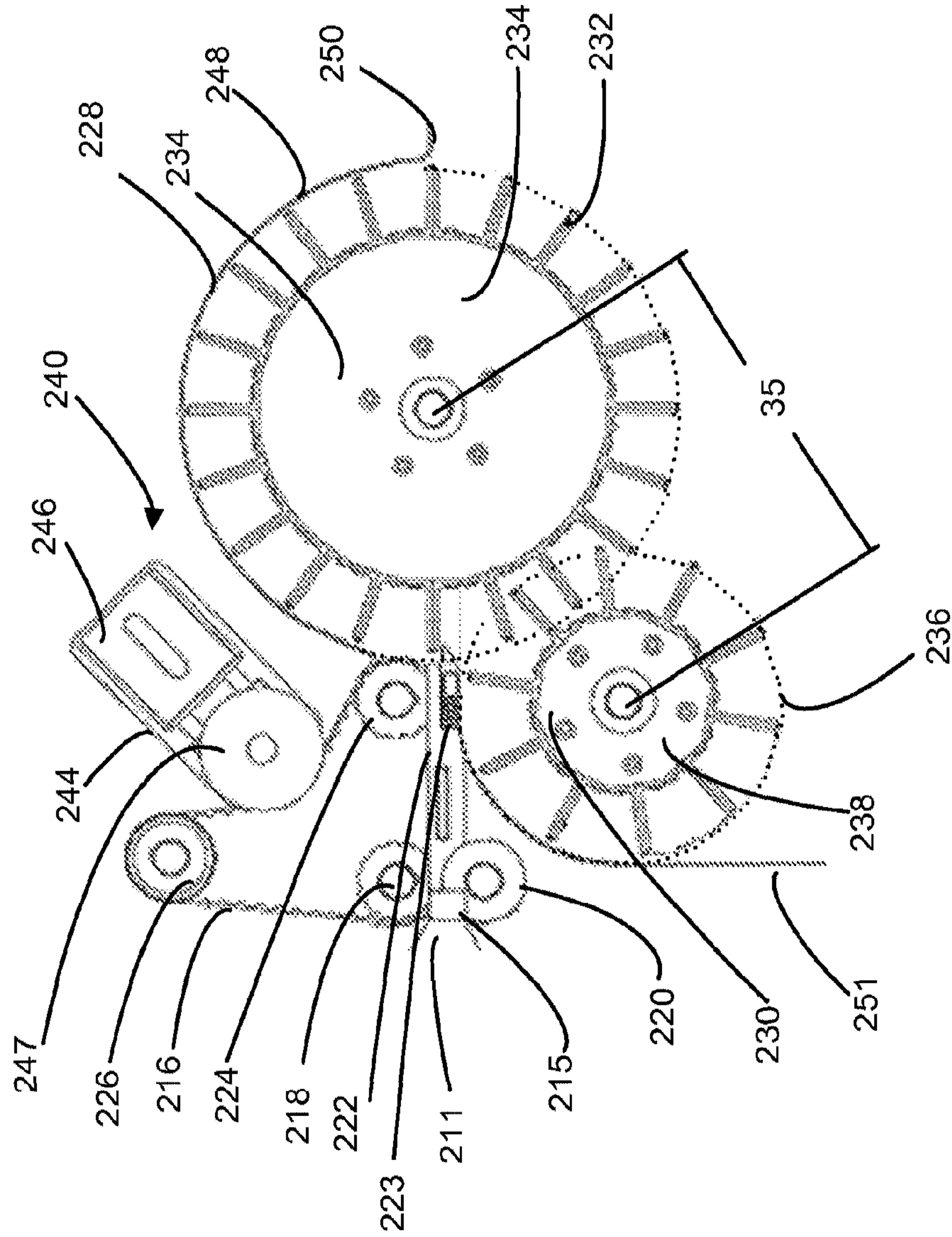


FIG. 15

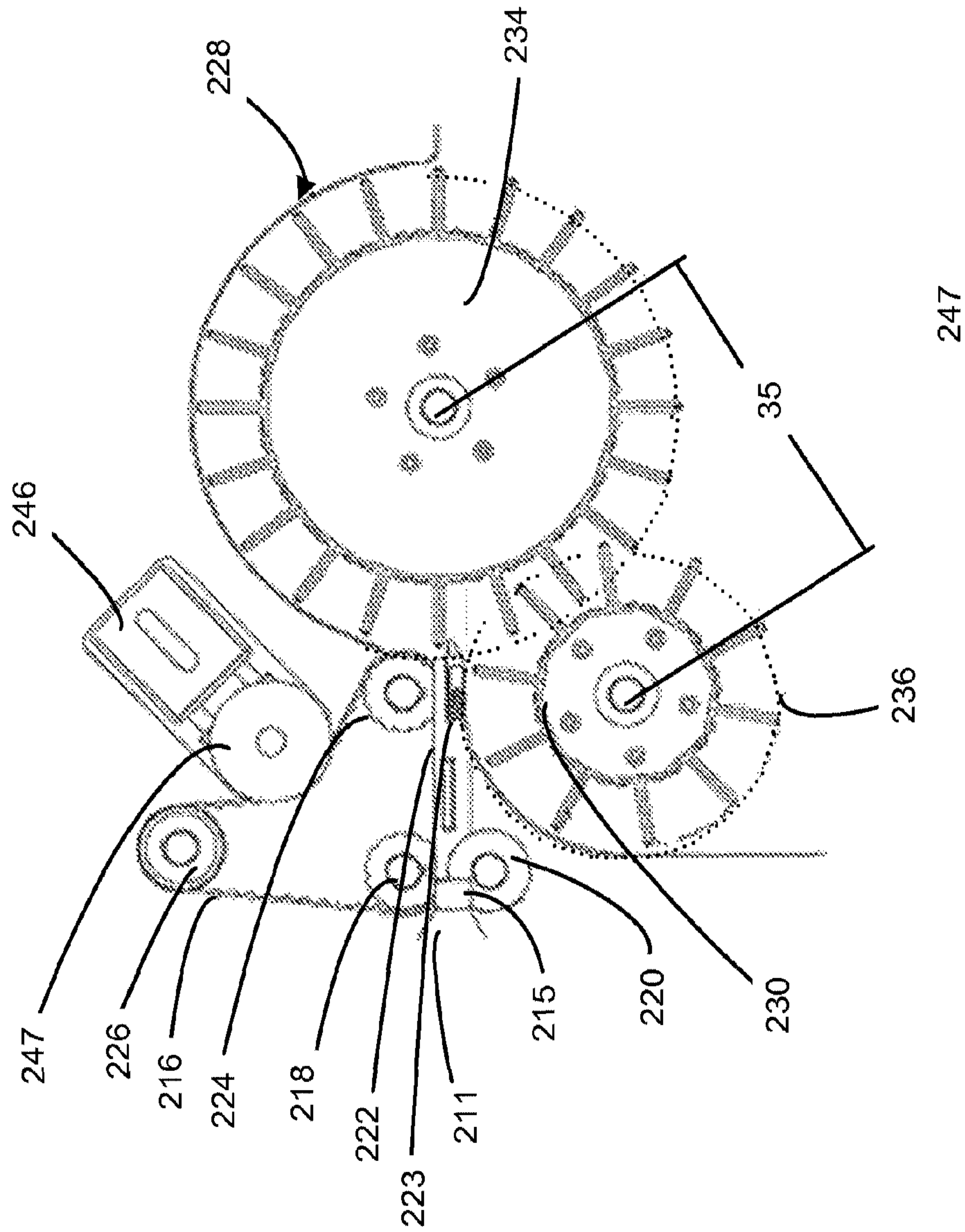


FIG. 16

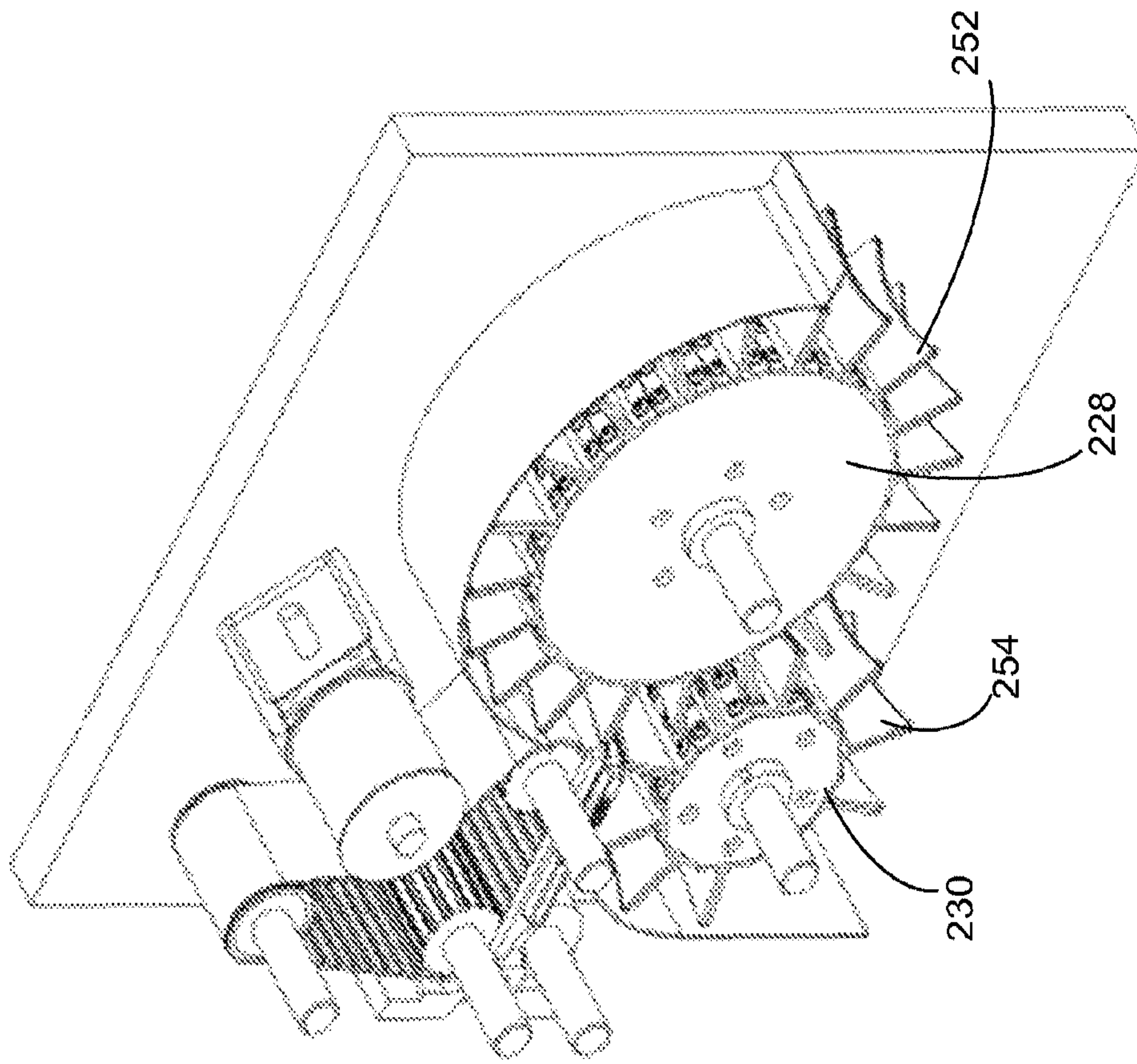


FIG. 17

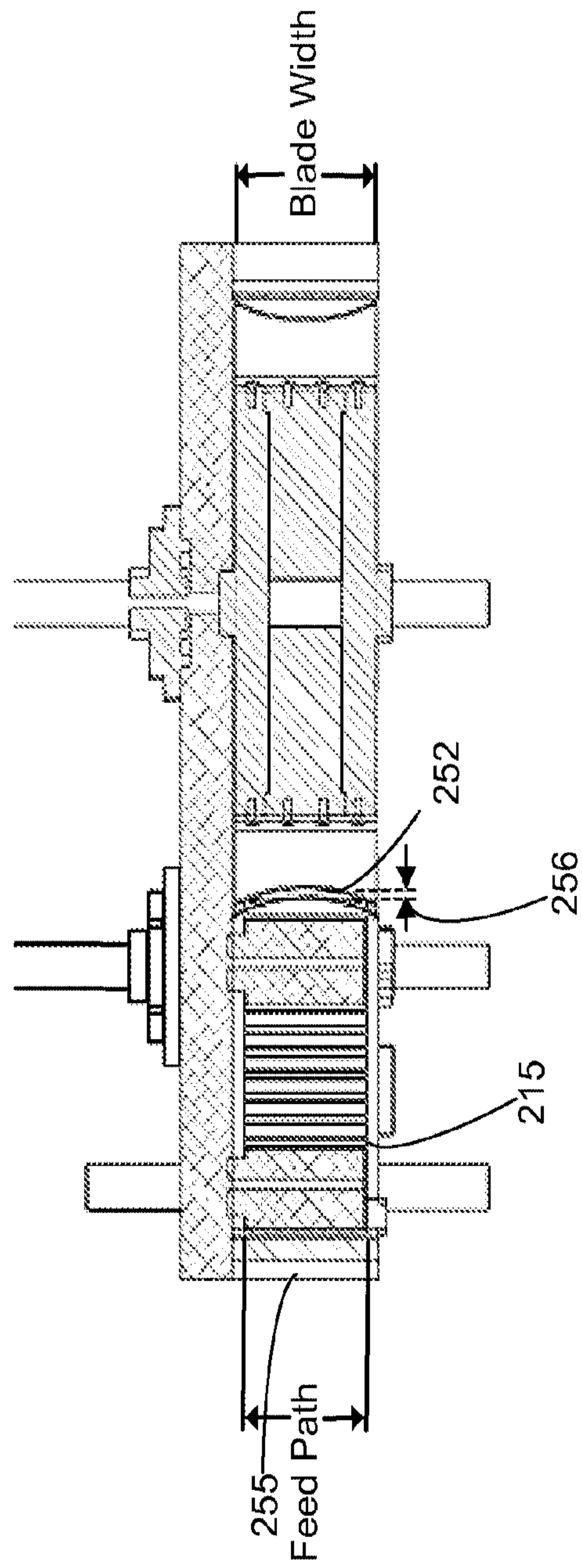


FIG. 18

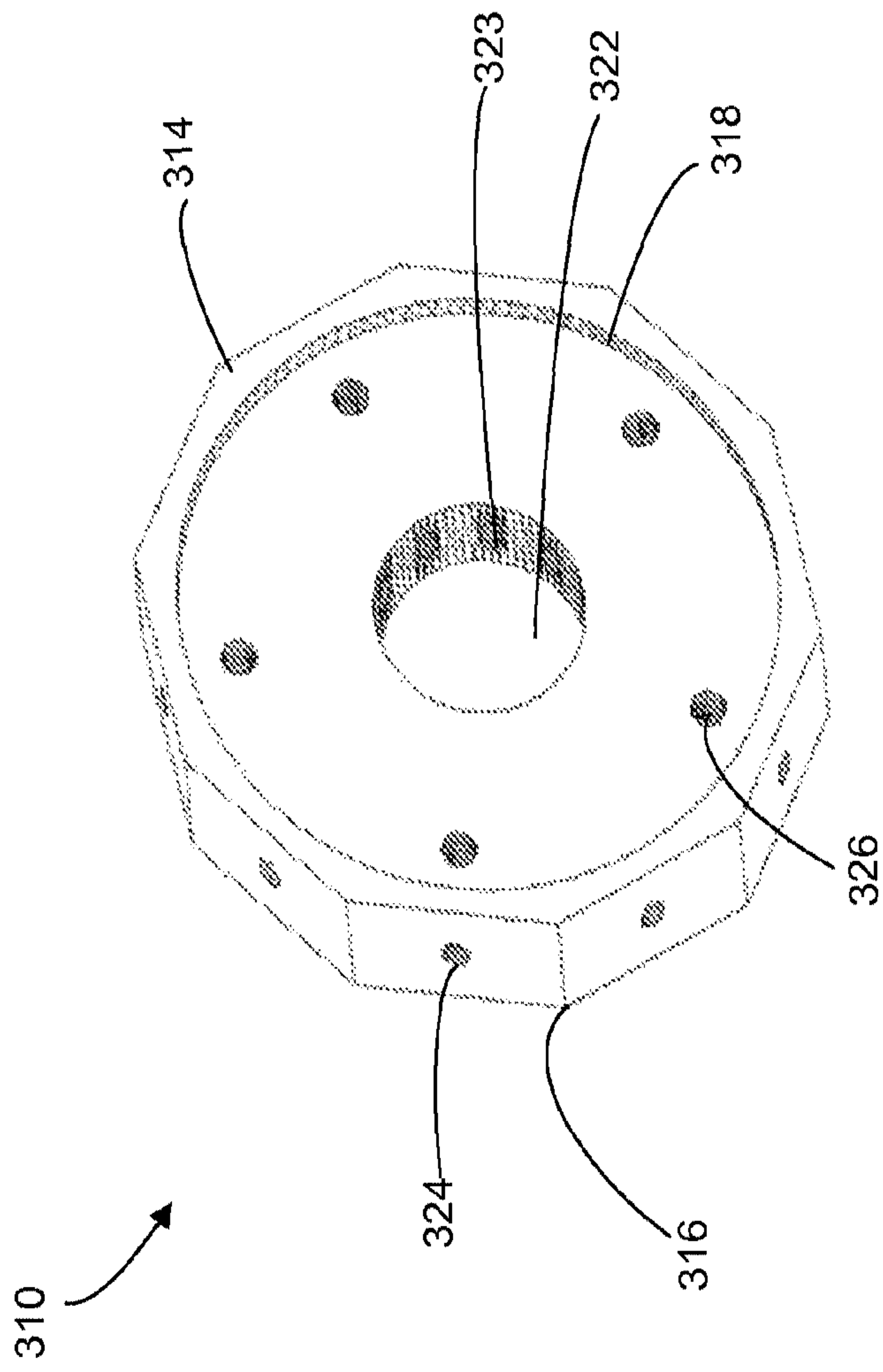


FIG. 19

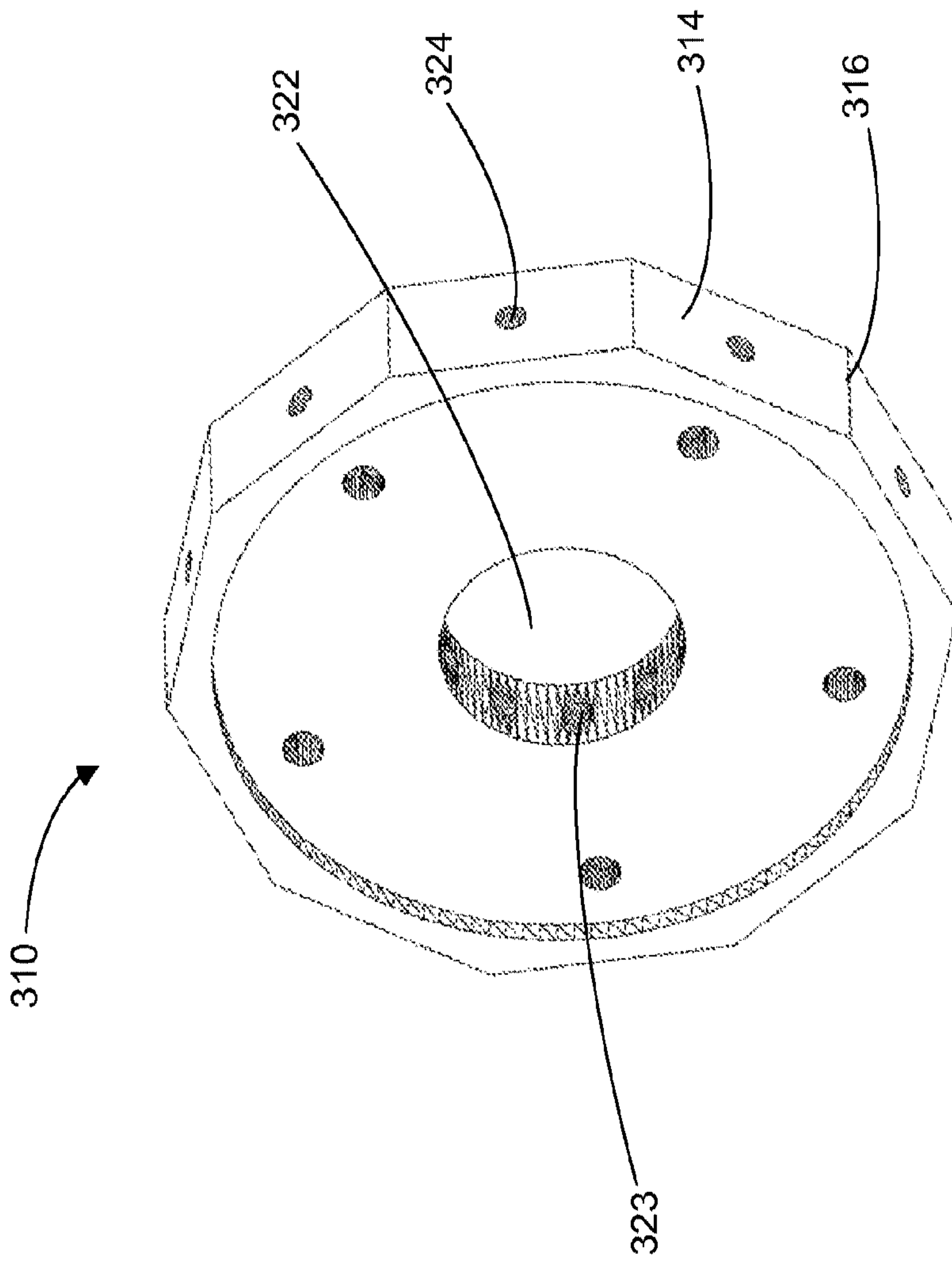


FIG. 20

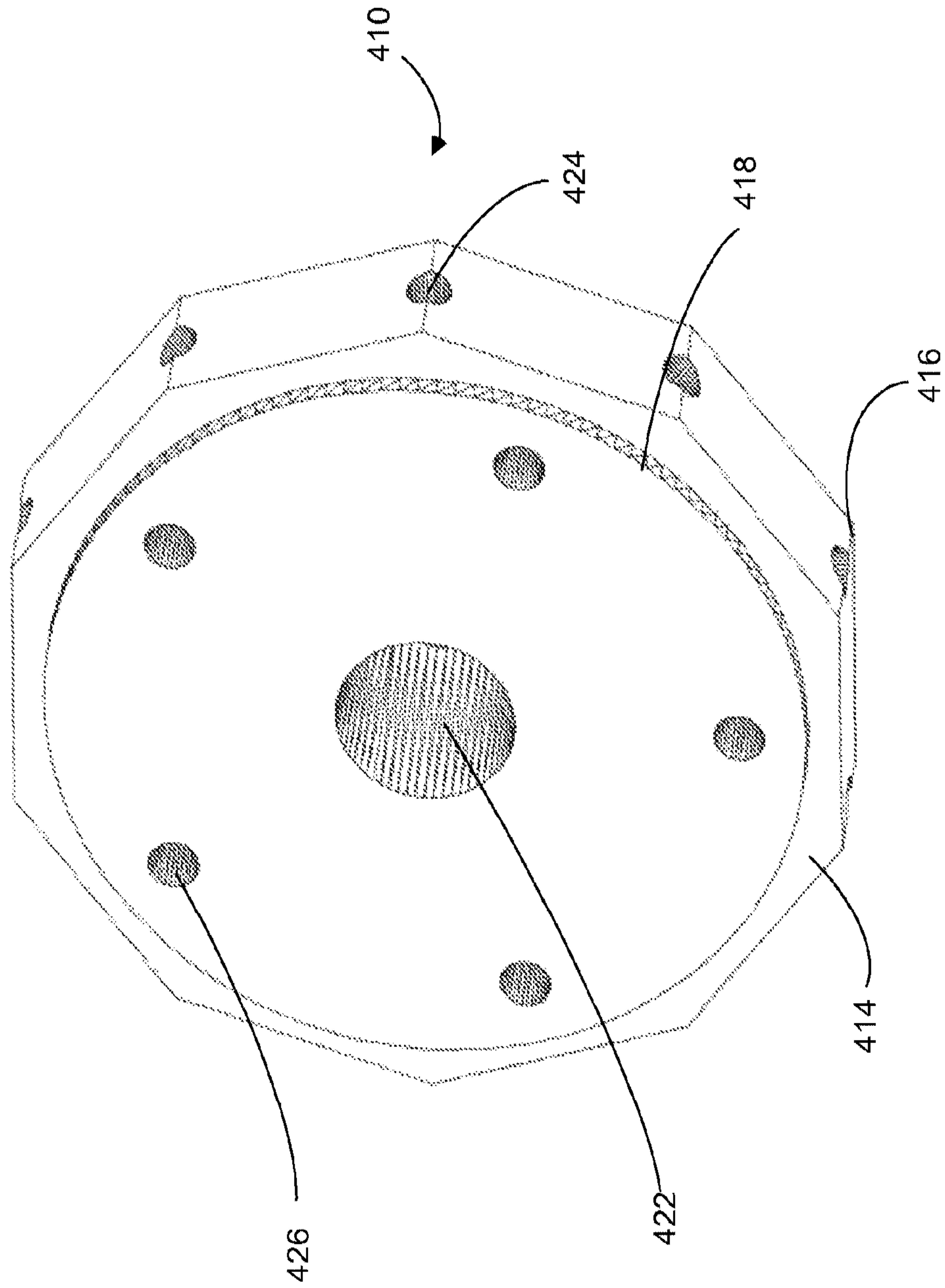


FIG. 21

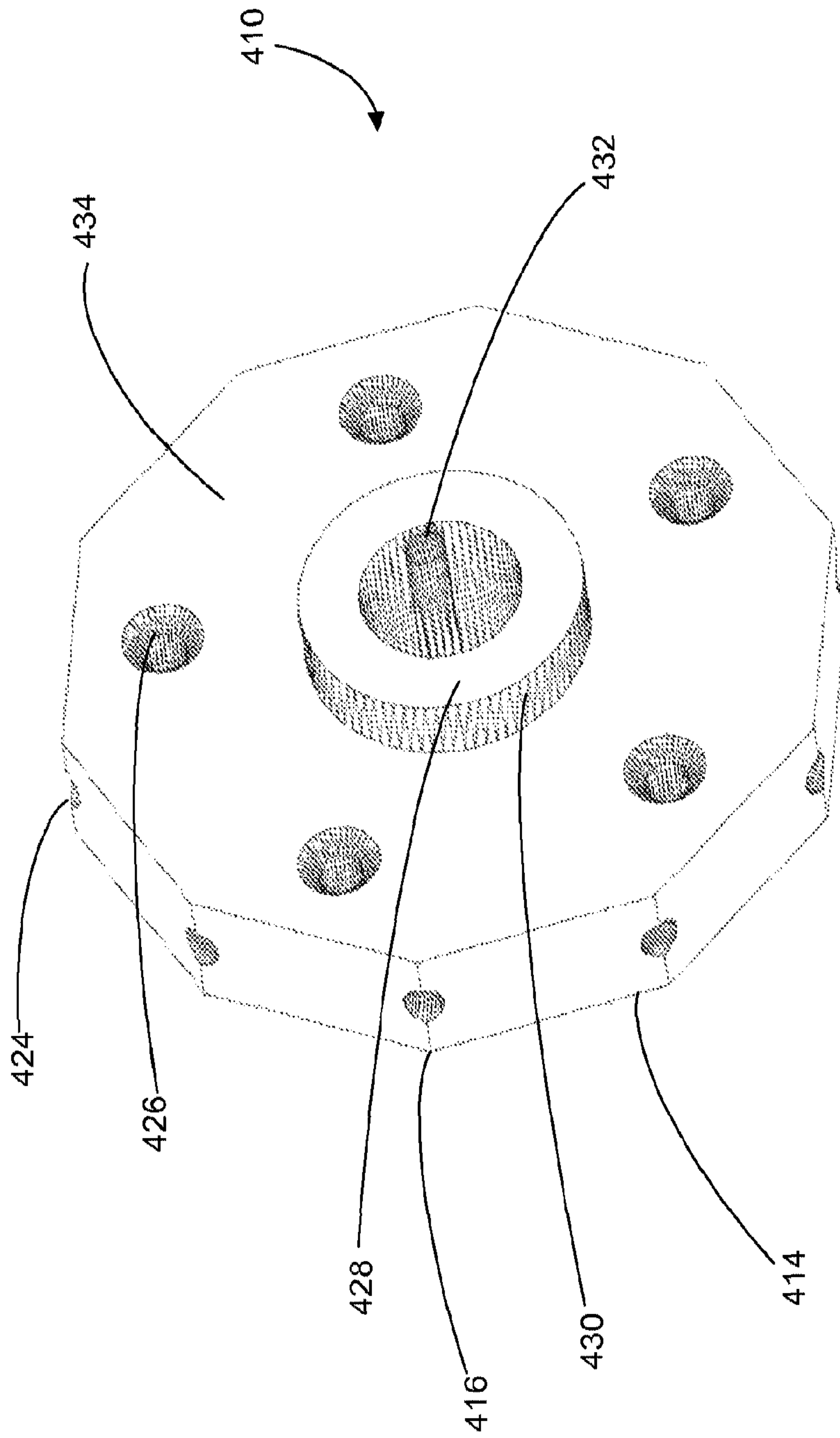


FIG. 22

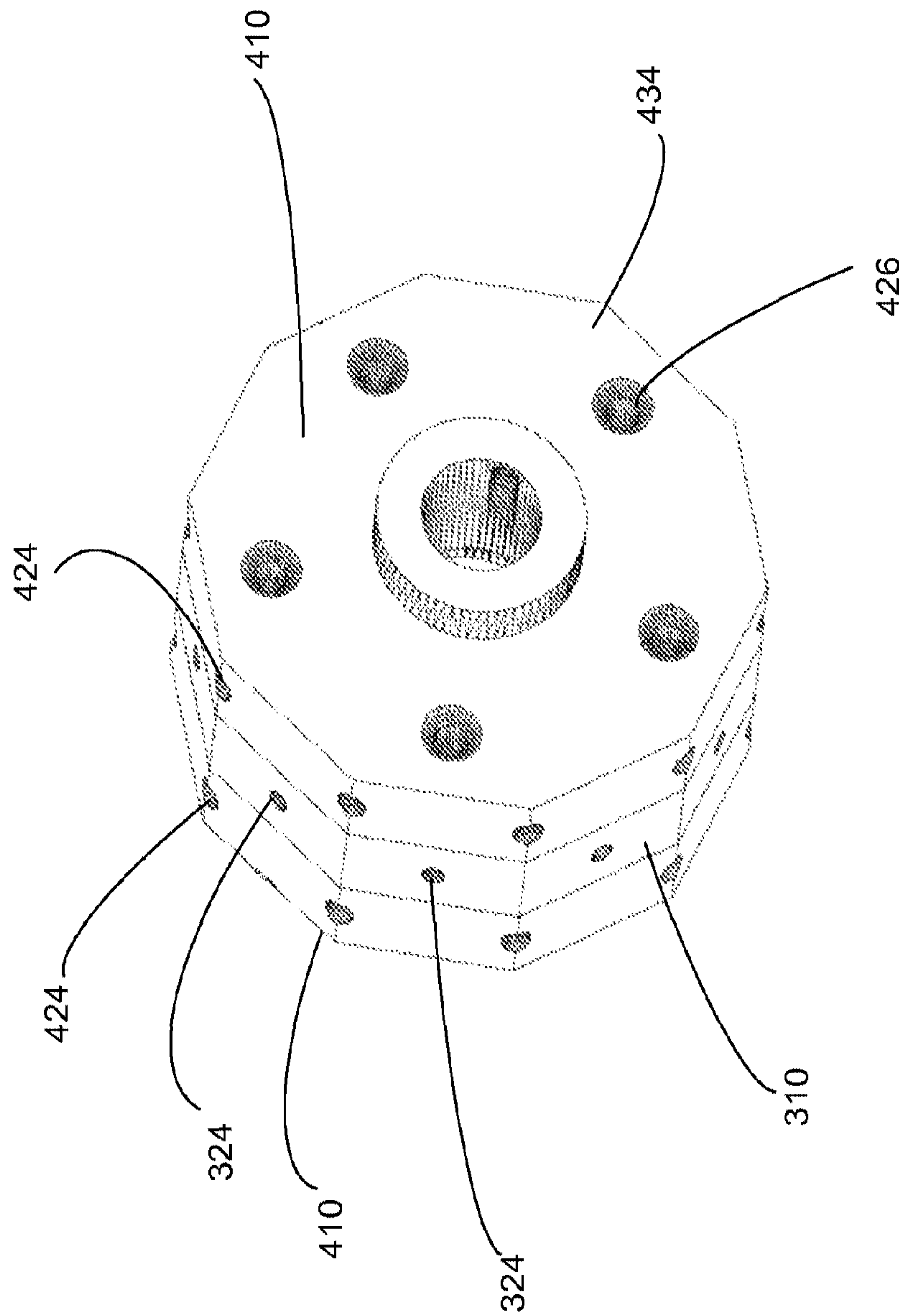


FIG. 23

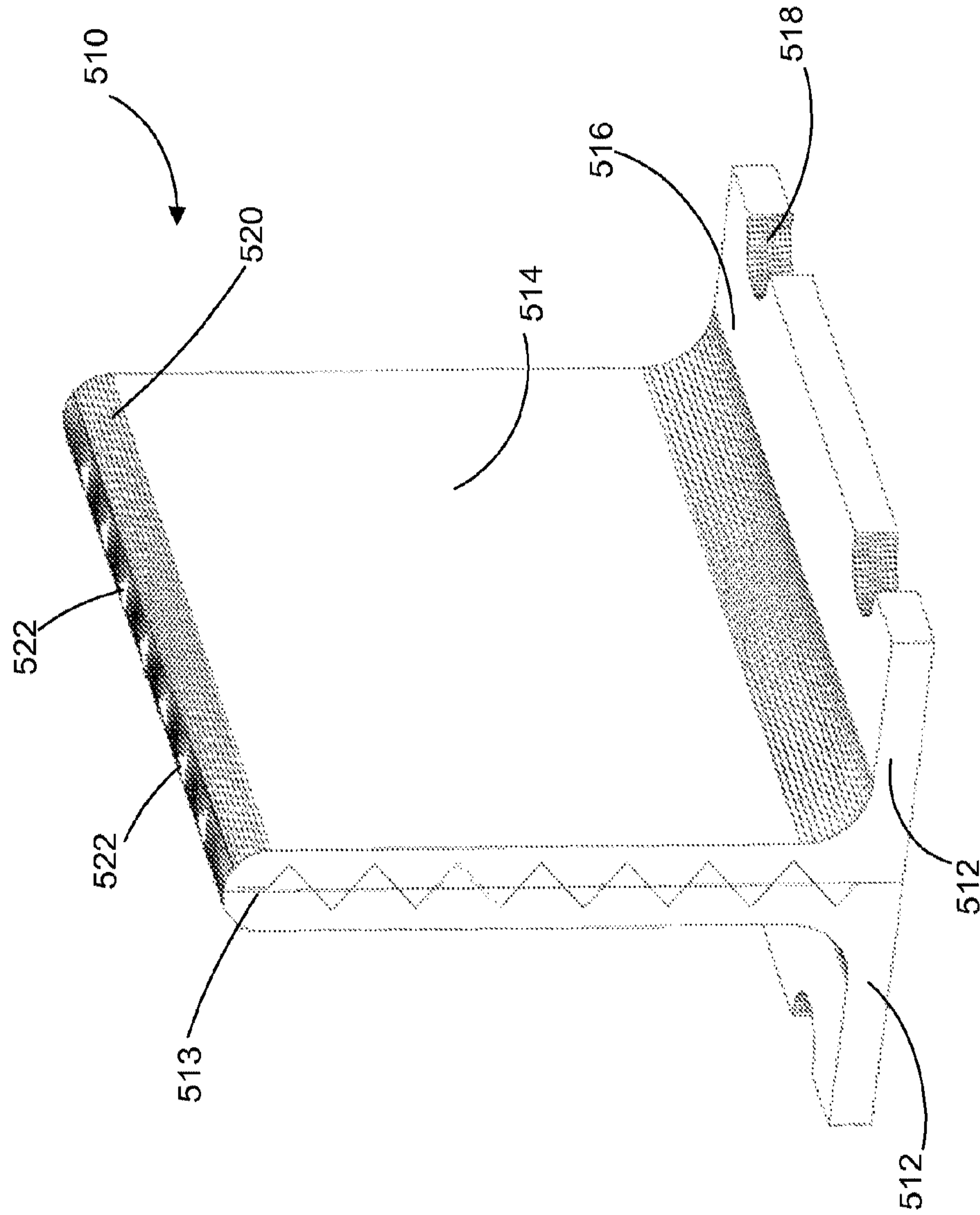


FIG. 24

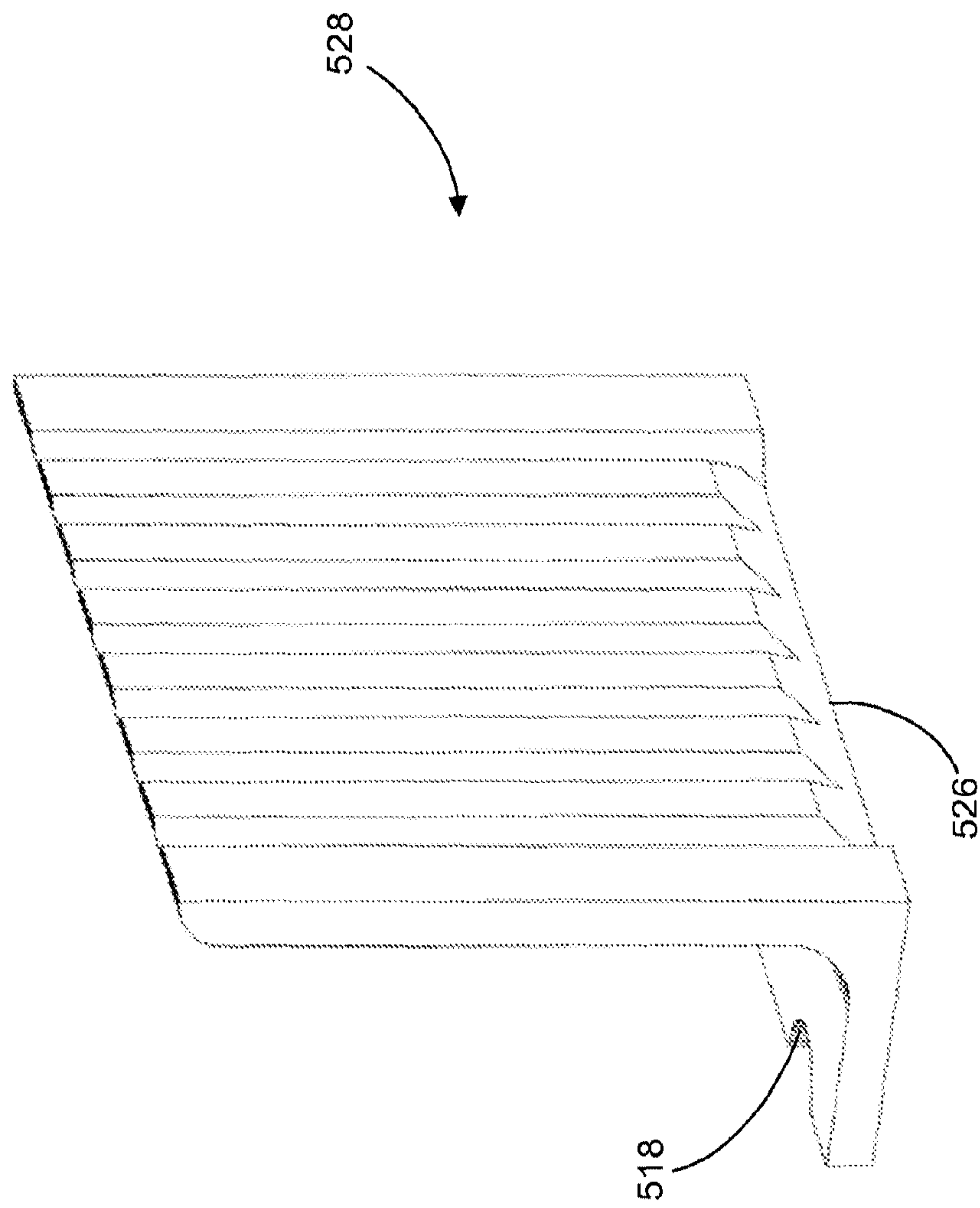


FIG. 25

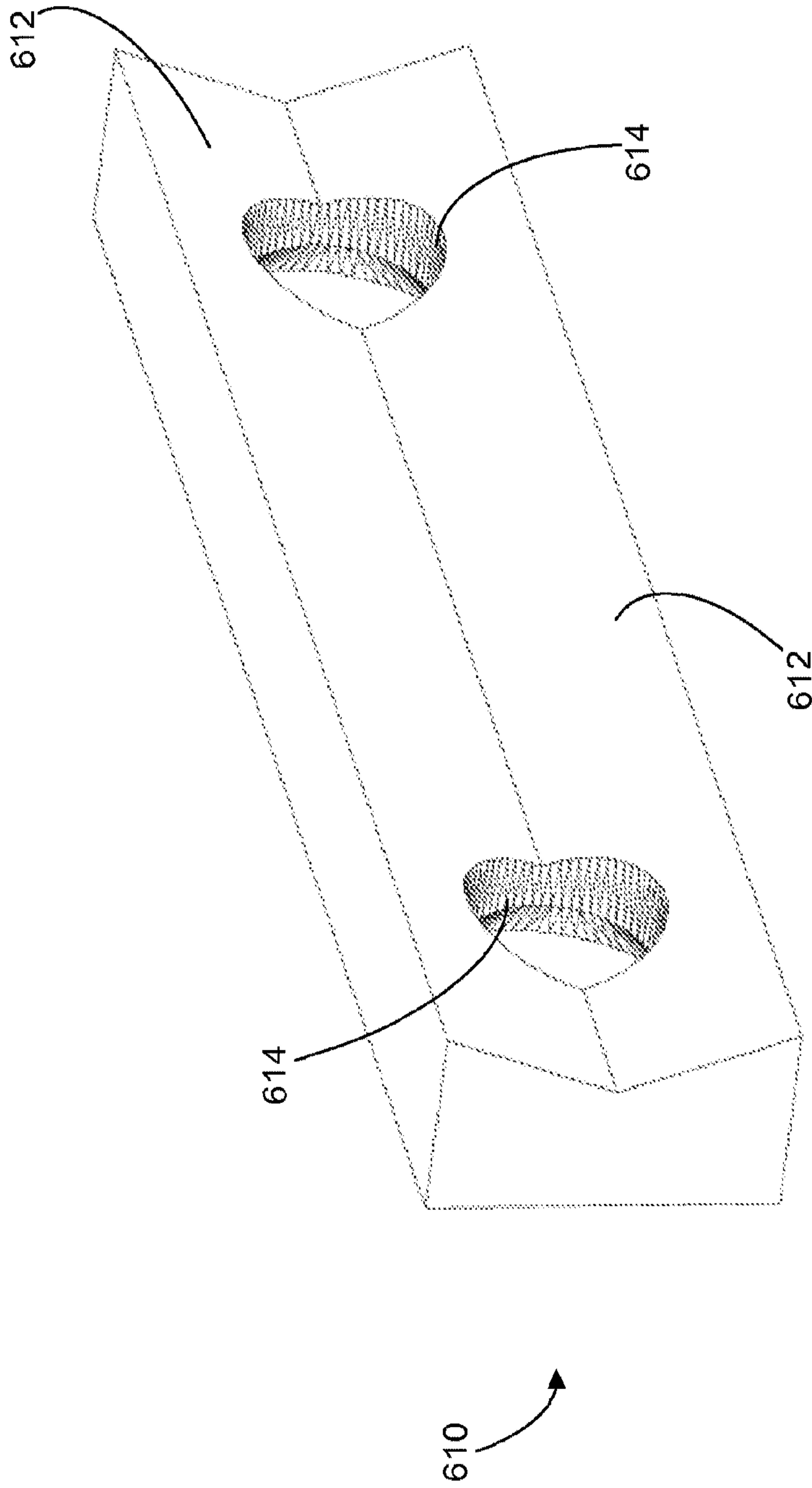


FIG. 26

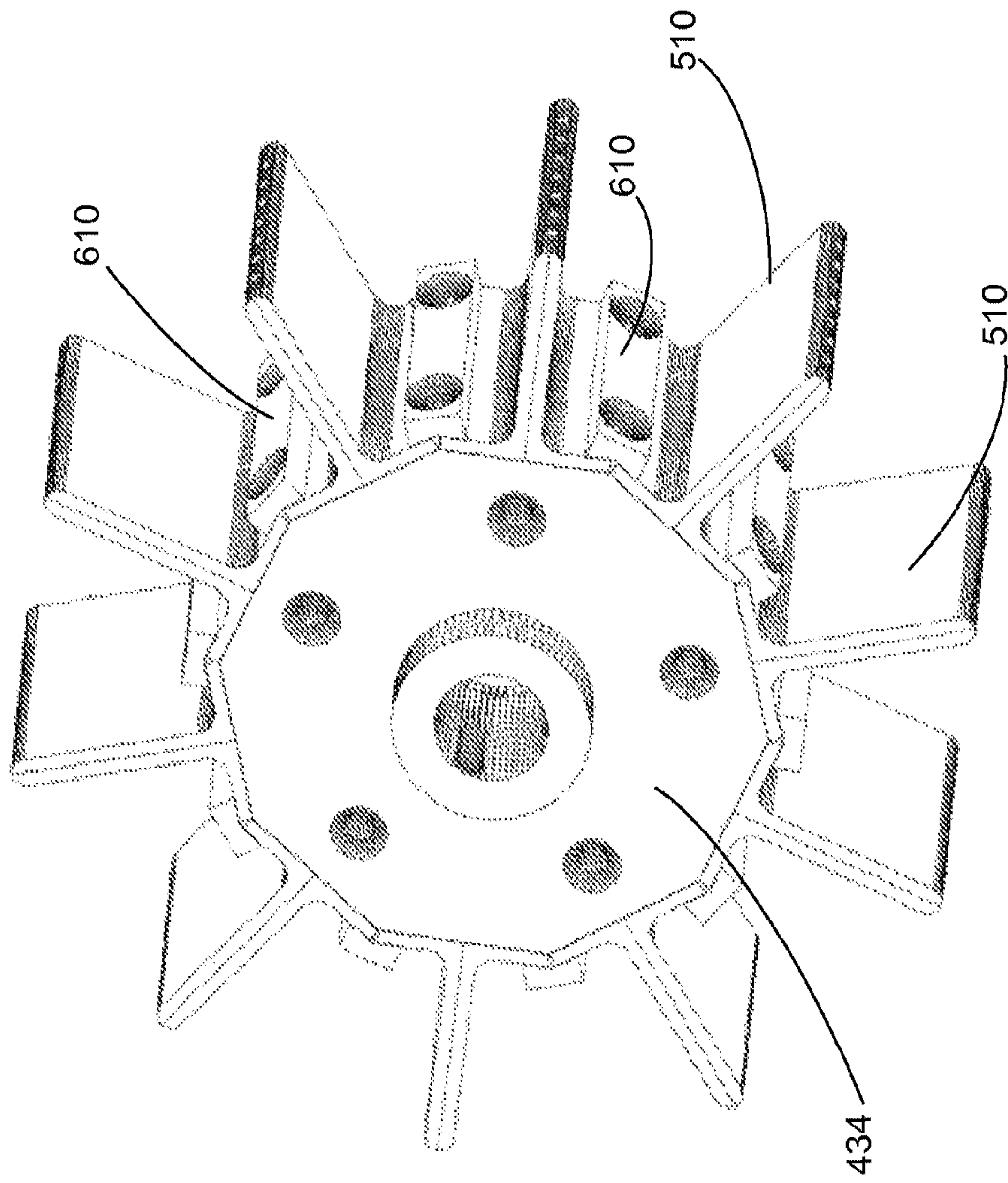


FIG. 27

FIBROUS PLANT STALK DECORTICATOR

Matter enclosed in heavy brackets [] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue; a claim printed with strikethrough indicates that the claim was canceled, disclaimed, or held invalid by a prior post-patent action or proceeding.

CROSS-REFERENCE TO RELATED APPLICATION

This is a reissue Of U.S. Pat. No. 9,957,642, issued on May 1, 2018 from U.S. application Ser. No. 14/733,676, which is a Continuation of U.S. application Ser. No. 13/807,047 filed as application No. PCT/US2011/042245 on Jun. 28, 2011, now U.S. Pat. No. 9,080,257, which further claims priority to U.S. Provisional Application No. 61/359,089 filed on Jun. 28, 2010. The entire contents of the above-identified applications are hereby incorporated by reference herein.

BACKGROUND

This disclosure relates to a decorticator, a type of machine designed to separate the outer bast or bark fiber from the inner woody core of plant stalks such as hemp, flax, kenaf, jute, nettles, and other plants in a single operation, by ways of an improved method of rapidly breaking the woody core of bast fiber stalks at very short intervals; rapidly scutching the finely broken core material from the bast fiber; and, accomplishing said action without causing mechanical damage to the bast fiber. This action is equally effective on plant stalks in wet, dry, unretted, or retted condition.

Past and present decortication machines, including but not limited to breakers, scutchers, hammermills, differential speed decorticators, and pre-disintegrators, invariably cause mechanical damage to bast fiber. This mechanical damage is commonly identified as "kink bands." Typically, the drier the stalk and lesser the degree of retting then the higher the instance of mechanical damage.

Most modern decortication systems operate on dry partially retted stalks. The kink bands substantially weaken the bast fiber, thereby limiting the range of end-use applications that it may be used. For example, when mechanically damaged bast fiber is subjected to wet processing, otherwise referred to as degumming, and, additional mechanical processing such as carding, fine cleaning, or cottonization, the mechanically damaged fiber breaks into short fiber under 10 mm in length. Fiber under 10 mm in length has little value and renders attempts to add value through further processing economically unfeasible.

A decortivating machine is described in U.S. Pat. No. 2,460,488, which employs a method of decortication that includes the following components 1) crushing rolls; 2) feed table; 3) gripping roll; 4) breaking roll; and, 5) scutching roll. The machine operates in the following manner.

Fibrous plant stalk material is fed longitudinally into two crushing rolls vertically aligned one on top of the other and made of smooth hardened metal. The crushing rolls rotate oppositely and drive the fibrous plant stalks onto a metal feed table which has a gripping roll made of hard rubber positioned above the feed table in close proximity to the feed table edge in order to hold the stalks while continuing to drive the stalks over the edge of the table into the orbit of the breaking roll and then the scutching roll.

Both the breaker and scutcher rolls have a plurality of solid metal blades extending outward longitudinally from the central shaft. The face of the blades is perpendicular to the directional flow of the fibrous materials being acted upon. The two rolls, breaker and scutcher, rotate in opposite directions and their respective blades intermesh in their respective orbital paths to a depth not greater than 1.5 inches.

The forward moving stalks are driven over the feed table edge into the orbital paths of the breaking and scutching rolls. The centers of the breaking and scutching rolls and feed table edge form approximate 45-45-90 triangle with the ninety degree angle formed between feed table edge and the centers of the breaking and scutch rolls.

The center of the breaking roll is positioned straight out from the feed table edge relatively in line with the plane of the feed table. The orbital path of the breaking roll is down and toward the edge of the feed table. The orbital path clears the edge of the feed table by less than an inch thereby impacting the forward moving stalk bending it down approximately ninety degrees across the feed table edge.

The forward moving stalk continues into the orbital path of the oppositely rotating scutcher roll, which has an orbital path that clears the bottom of the feed table by less than an inch. The blades of the scutcher roll impact the forward moving stalk at substantially the same point of impact as the preceding impact with the breaking roll, but bending the stalk back and up at approximately ninety degrees. The method of decortication involves rapid back and forth bending which breaks, loosens, and allows the bast fiber to be cleaned of woody core without causing serious damage to the fiber. It also allows for operation on plant stalks of varying degrees of moisture content. This method has antecedent in U.S. Pat. No. 2,980, as well as U.S. Pat. No. 1,601,984.

Another decortivating machine is described in U.S. Pat. No. 2,162,780. In this particular decorticator, the breaking and scutching rolls are set at approximately forty five degree angles to the path of the fibrous materials, which are pulled through the breaker and scutcher rolls horizontally at a latitudinal or diagonal angle rather than longitudinally. This prior art also provides for ways of arranging sections of blade in a regularly recurrent offset pattern longitudinally across the breaker and scutcher rolls, as well as for ways of directing a stream of air from the central shaft of the top breaker roll from the base of the trailing faces of the breaker blades.

The previous prior art examples fail to provide or address the following points of innovation. A way of feeding and compressing a mat of plant stalks into the machine through feed belts position under and above the path of travel. A way of adjusting the angle of the feed table in such a manner that the distance between the point that the stalks clear the feed roller's grip and enter the orbit of the breaker roll can be controlled for optimal performance. A way of controlling the lateral diffusion of plant stalks between the scutcher rolls and feed roller with lateral baffle plates. A way of controlling the lateral diffusion of the decorticated fiber through the zone of decortication by providing concave blade tips. A decortication element configuration consisting of a larger upper breaking roll and a smaller lower scutching roll, which results in a unique kinematic action embodied in a faster traveling lower scutcher that sweeps core from fiber. Significantly this sweeping action may be optimized by altering the pitch point of the lower scutcher in relation to the orbit

of the upper breaker resulting in lesser or greater distances between the blades of the scutcher and breaker throughout their orbital intermeshing.

A way of directing jets of air through the breaking and scutching blades at the point of contact with the fibrous material.

The smoothing and rounding of the feed table edge, breaking blades, and scutching blades.

The application of slick hydrophobic coatings and finishes to the feed table, feed table edge, breaking blades, and scutching blades.

The addition of longitudinal combing grooves in the feed table edge, breaking blades, and scutching blades that are directionally aligned parallel to the path of travel of the fibrous materials.

A way of controlling the operational velocity relative to impact action of the decortication elements by providing a separate variable speed power drive for the feed system.

An optimal configuration of crushing elements in relation to gripping element and decortication element, i.e., feed table edge, breaking roll, and scutching rolls.

A simplified design for breaking and scutching roll lateral extension, based on a central spacer and two hubs that allow for multiple blade configurations, such as offset, helical, or straight.

SUMMARY

A first object of this invention is to provide novel pneumatic elements hereafter referred to as an "air comb" that are built into each breaking and scutching blade and capable of delivering a plurality of forceful jets of air at the point of contact with the fibrous material and throughout the orbital path from the tip or extremity of the blades in order to more effectively open fibrous materials, remove core, decrease intermeshing of blades, reduce fiber damage, increase throughput, remove moisture, prevent winding of fiber, and generally aid in material handling throughout the process of decortication.

A particular object of the air comb feature is to provide a way of delivering chilled air to enhance the effectiveness of core removal. Another particular object of the air comb is to provide a way of delivering heated air to enhance moisture elimination.

A particular object of the air comb blade is the concaved tip designed to prevent the fibrous material from escaping outside the lateral periphery of the air comb blades by providing a natural space for the fibrous material to fill into as it is acted upon during the course of decortication.

A second object of this invention is to provide a novel decortication element arrangement consisting of an upper breaking roll with a greater outer diameter than the lower scutching roll. As a result of this arrangement the lower scutching roll must travel at a proportionally faster peripheral speed than the upper breaking roll causing a unique kinematic action which can be adjusted by altering the outer diameter of the scutching roll or by moving the pitch point of the scutching roll blades in relation to the orbital path of the breaking roll blades. The overall effect of this kinematic action creates a highly beneficial cleaning action throughout the orbital intermeshing of the breaking and scutching roll blades. This kinematic action is best described as a sweeping action accompanied by additional bending and flexing of the fibrous material. The end result is cleaner fiber.

The third object of this invention incorporates or applies physically slick and hydrophobic resins, composite finishes,

coatings, or materials to the air comb blades, feed table, and feed table edge as novel way to improve flow ability of fibrous materials, reduce frictions, and eliminate fiber damage.

The fourth object of this invention provides for round and smooth edges on the feed table and air comb blades in order to minimize fiber damage.

The fifth object of this invention is to cut longitudinally aligned combing grooves running parallel to the path of travel of the fibrous materials in to the rounded and smoothed edges of the feed table edge and air comb blades in order to more effectively open fiber bundles and clean core.

The sixth object of this invention provides a breaker and scutcher assembly that will allow for ease of lateral extension with a central spacer and end hubs. Said arrangement also allows for multiple blade geometries such as symmetrically offset, straight, or helically spirally patterns. Said arrangement also provides for a way of locking blades in place.

The seventh object of this invention is to calibrate throughput velocity in relation to decortication impacts by way of separate variable speed drives for the feed element and decortication element.

The eighth object of this invention provides a way for adjusting the angle of the feed table from 0 degrees up to 10 degrees in order to the increase or decrease the distance between the point of contact between the feed roller and the feed table in relation to the orbit of the upper breaking roll.

The ninth object of this invention provides a way for adjusting the length of the feed table in order to increase or decrease the distance between the edge of the feed table and the orbit of the breaking.

The tenth object of this invention provides a novel feed element consisting of lower belt and upper belt which compress plant stalks into crushing rolls and continue to feed fibrous material over a feed table and under a gripping feed roll thereby creating a uniform continuous mat of material to be acted up in the decortication element.

The eleventh object of this invention provides a lateral baffle or guide between the crushing rolls and through the gripping feed roll in order to compress and guide the fibrous material into the action of the decortication elements.

BRIEF DESCRIPTION OF THE DRAWINGS

It is possible to thoroughly understand this invention and its benefits by referring to the following description of operating parts in relation to the drawings and subsequent explanation of the preferred method of operation.

FIG. 1 is a side elevation view of a fibrous plant stalk being driven through a two dimensional side perspective of the decortication element.

FIG. 2 is a three dimensional obtuse overview of the same operational configuration of the feed mechanism and decortication element.

FIG. 3 shows a way of constructing a polygonal rim segment for the breaking and scutching rolls with exterior ports and channels to the hollow interior.

FIGS. 4, 5, 6, and 7 are perspective views of an air comb.

FIGS. 8, 9, and 10 are exploded perspectives of one side from the exterior to the center of the machine.

FIGS. 11, 12, and 13 are exploded perspectives of the opposite side from the exterior to the center of the machine.

FIG. 14 is a side view of a fibrous plant stalk decorticator.

FIG. 15 is a side view of another fibrous plant stalk decorticator.

5

FIG. 16 is a side view of another fibrous plant stalk decorticator having an adjustable feed table.

FIG. 17 is a perspective view of a fibrous plant stalk decorticator.

FIG. 18 is a top view perspective of a fibrous plant stalk decorticator.

FIG. 19 is a perspective view of a portion of a hub assembly.

FIG. 20 is another perspective view of the portion of the hub assembly of FIG. 18.

FIG. 21 is a perspective view of a portion of a hub assembly.

FIG. 22 is another perspective view of the portion of the hub assembly of FIG. 20.

FIG. 23 is a perspective view of an assembled hub.

FIG. 24 is a perspective view of a decorticator blade.

FIG. 25 is a perspective view of an assembly component for the decorticator blade of FIG. 23.

FIG. 26 is a perspective view of a clamp for assembling the blade of FIG. 23 to the hub of FIG. 22.

FIG. 27 is a perspective view of a hub and blade assembly.

DETAILED DESCRIPTION

The machine illustrated in FIGS. 1-2 and FIGS. 8-13 is mounted within a pair of solid upright frame structures indicated at 14 (FIGS. 8-13). The frame structures 14 are adequately spaced to accommodate the width of the feeding mechanism 2a, 2b, 3, and 13, and decortication elements 5, 6, and 7. The machine is supported by triangular legs 15. There is an input port at 13 and output port at 35. It should be readily evident from these drawings that said invention essentially represents a box which may be framed, mounted, and supported by a multitude of structural elements. Specifically such box designs are ideal for implementation into forage and combine harvesting equipment.

There are a multitude of ways available for feeding the stalks into the influence of the crushing rolls 2a and 2b. The machine illustrated in FIGS. 8-13 is fed by manual ways via input port 13.

FIG. 1 shows fibrous plant stalk 1 being fed into the oppositely rotating crushing rolls 2a and 2b, which are mounted on shafts 16 and 18. In the drawings, FIGS. 8 and 9, the shafts 16 and 18 fit into bearings 17a-17b and 19a-19b that are supported in structural frame 14. FIGS. 11 and 12 show a manner in which crushing rolls 2a and 2b are driven in tandem with gripping roll 4 by way of drive shafts 16, 18, and 20, which attach to their respective sprockets 38, 39, and 36 and are propelled by drive shaft 22, which attaches to sprocket 37, and drives chain 36.

In the preferred embodiment of this invention, which is not shown in the drawings, but readily understandable, bearings 17a and 17b should be attached to suitable structural guides on both sides of the structural frame 14. These structural guides support bearings 17a and 17b and allow vertical movement. These structural guides will allow suitable pressure yielding devices such as coil springs to be mounted in the structural frame 14 connected to bearings 17a and 17b in order to compress crushing roll 2a into crushing roll 2b.

In FIG. 1, fibrous plant stalk 1 is crushed between crushing rolls 2a and 2b as it is propelled forward by the counter-clockwise rotation of crushing roll 2a and the clockwise rotation of crushing roll 2b. This flattening action serves to create a uniform layer of fibrous stalk material and begins to break down the bonds between the bast fiber and

6

woody core. In addition, the squeezing action removes moisture from wet plant stalks.

Upon exiting the influence of crushing rolls 2a and 2b, fibrous plant stalk 1 is received on feed table 3 which is relatively thin and made of smooth metal or composite material. Feed table 3 is supported in structural frame 14. It is beneficial to apply a slick hydrophobic coating or finish to feed table 3 in order to reduce friction and improve flow ability of fibrous plant stalk 1. It is also beneficial to perforate the surface of feed table 3, which similarly reduces friction and improves flow ability by providing moisture an escape route and air by blowing up through the holes due to the fanning action of scutching roll 7 and its attendant air combs 8.

In the preferred embodiment of this invention gripping roll 4 is made of hardened rubber, polyurethane, or similar gripping polymer and positioned directly above feed table 3 and directly adjacent to crushing rolls 2a and 2b along the path of travel of fibrous plant stalk 1. This compact design enables fibrous plant stalk 1 to be held under the dual influence of crushing rolls 2a and 2b, as well as, gripping roll 4 for as long a duration as possible thereby minimizing the chance of fibrous plant stalk 1 slipping into the decortication elements. This design also lends itself to uniquely optimized working diameters for crushing rolls 2a and 2b based on feed velocity and in relation to the diameter of scutching roll 7 and breaking roll 6.

In the drawings, gripping roll 4 is mounted on shaft 20, which is supported in structural frame 14 by bearings 21a and 21b. However, it is the preferred embodiment of this invention that gripping roll 4 also be mounted on vertically moveable bearings attached to structural guides that are built into structural frame 14 in order to engage a pressure applying device, such as a spring coil. Such a pressure device would be mounted to structural frame 14 and attached to bearings 21a and 21b, in order to compress gripping roll 4 against feed table 2 and firmly hold fibrous plant stalk 1.

Gripping roll 4 rotates counter-clockwise at a slightly faster velocity than crushing rolls 2a and 2b, pulling the stalk forward and not allowing it bunch up as it is fed into the influence of the decortication element.

Upon exiting the grasp of gripping roll 4, fibrous plant stalk 1 is propelled forward over feed table edge 5 and into the orbital paths of breaking roll 6 and then scutching roll 7. Feed table 3 is extendable and retractable allowing feed table edge 5 to be moved in and out to accommodate fibrous plant stalks of varying stalk diameter, flexibility, and brittleness. In general feed table edge 5 does not exceed the furthest circumferential point forward of gripping roll 4. This tangential point is perpendicular to feed table 3. The distance between feed table edge 5 and the peripheral orbits of breaking roll 6 is adjustable within a range of 5 millimeters to 44 millimeters.

Breaking roll 6 and scutching roll 7 may be arranged in numerous manners with varying diameters and numbers of blades. However in the preferred embodiment of this invention, breaking roll 6 and scutching roll 7 are the same size and have the same number of blades. As can be seen in FIGS. 1-2 and FIGS. 8-13, this symmetrical design necessitates that breaking roll drive shaft 28 intersect with the plane of feed table 3 and that scutching roll drive shaft 25 intersect with a tangential line dropped down from rounded feed table edge 5 that is relatively perpendicular to the directional flow of fibrous plant stalk 1 and in relative proximity to and behind feed table edge 5. Such an arrangement of working parts forms an approximate 90-45-45

degree triangle between feed table edge **5**, breaking roll drive shaft **28**, and scutching roll drive shaft **25**. The 90 degree angle occurs between feed table edge **5** and break roll **6** and scutch roll **7**. This arrangement also insures that breaking roll **6** and scutching roll **7** impact fibrous plant stalk **1** at substantially the same location. As a result of this action fibrous plant stalk **1** is repeatedly bent back and forth causing the core to be thoroughly shattered and removed from fibrous plant stalk **1**, I a, and I e.

FIGS. **8** and **11** show means for securing breaking roll **6** and scutching roll **7** to both sides of structural frame **14**. Specifically, drive shafts **25** and **28** correspond to breaking roll **6** and scutching roll **7** respectively are secured in bearings **26a-26b** and **29a-29b** respectively. Both sets of bearings, **26a-26b** and **29a-29b**, may be mounted on swivels that are secured to structural frame **14**, in order to better control the angle of impact and depth of intermeshing between air comb blades **8** on the opposing rolls **6** and **7**.

FIGS. **1** and **2** depict beater roll **6** and scutcher roll **7** with a plurality of blades **8** of special construction, hereafter referred to as air combs. The preferred design for these air combs is illustrated in FIGS. **4-7**. FIGS. **4** and **5** show a single L-shaped piece of machined metal with grooves cut into the back exterior side **8a** and a rounded inner tip at the top **8b**. When two L-shaped pieces are placed back to back the resulting composite piece is an air comb **8** as shown in FIGS. **6** and **7**.

Air combs attach to breaking roll rim segment **9** and scutching roll rim segment **9**, which are shown in FIGS. **1** and **3**. FIG. **3** also shows the ports into which air combs **8** plug into rim segments **9**. FIG. **3** shows how these ports open into hollow shaft interior **10** which is also depicted in FIG. **1**. Compressed air is delivered to hollow shaft interior **10** via drive shafts **25** and **28** on the opposite side as the drive mechanisms, which in the present drawings corresponds to FIG. **11**. This air may be chilled or heated. In the preferred embodiment of this invention the compressed air is channeled through volume reducing grooves which effectively accelerate the air stream into jet blasts more effectively opening fiber bundles **1a** and removing core **1c**. The air combs also reduce the angles of impact and degree of intermeshing between the blades of breaking roll **6** and scutching roll **7**. Consequently, less impact force is required to remove the core **1c** from flattened and decorticated fibrous stalk material **1a**, which minimizes mechanical fiber damage to recovered fiber **1b**. The "air comb" as described in FIGS. **4-7** is an entirely novel device. However, it is not limited to this design. An air comb **8** may be simply constructed from a typical air knife or similar volume reducing jet nozzle type device that is capable of delivering pressurized air at the point of contact with the fibrous material and throughout the orbital path from the tip or extremity of the blades. Such air knives and nozzles are available on the open market. To make a suitable air comb with one of these devices one would reinforce the forward travelling side of said device with a suitable hardened material such as metal or composite to form a blade for breaking and scutching operations. An air knife, jet, or plurality of jets reinforced in such a manner is novel and the proposed application of such a device is novel.

Another feature with benefits that is not shown in the present drawings is combing grooves aligned parallel to the directional flow of fibrous plant stalks cut into feed table edge **5** and air comb **8**. Combing grooves facilitate the opening of fiber bundles, which improves core removal Ie.

Feed table edge **5** and air comb **8** are all rounded and smooth finished, even with combing grooves. The surfaces of these parts also benefit from the application of slick

hydrophobic coatings and finishes that reduce friction and improve the flow ability of fibrous plant stalks. These design measure allow fiber bundles to slip across impacting surfaces reducing the potential for mechanical damage.

Clean bast fiber **1b** is ejected from the decorticator through output port **35**.

A way of making rim segment **9** is shown in FIG. **3**. Rim segment **9** is manufactured from a cylinder or symmetrical polygonal shaft or pipe composed of metal or suitable hardened composite material. Rim segment **9** is constructed with ports in the outer surface to accept air comb blades **8**. These ports breach a hollow shaft interior **10** and permit air to enter the air comb **8**. Rim segment **9** is attached to the drive shafts **25** and **28** at the lateral ends of rim segment **9** by way of solid plate **12**, FIG. **2**, thereby creating sealed hollow shaft interior **10** which is capable of accepting pressurized air.

Rim segment **9** may be constructed as a single segment machine as depicted in the present drawings or rim segment **9** may be interlocked with multiple rim segment **9** pieces to create breaking and scutching rolls **6** and **7** of varying widths. When rim segment **9** is interlocked to form a multi-segment roll it is possible to arrange the blades in patterns, such as straight lines or as recurring offset patterns, like chevrons or helical spirals.

Elements for propelling breaking and scutching rolls **6** and **7** are illustrated in FIGS. **8** and **9**. Electric motor **34** drives chain **31** which loops around sprockets **27** and **30** propelling drive shafts **25** and **27** which correspond to breaking roll **6** and scutching roll **7** respectively. Chain **31** also drives sprocket **24** propelling drive shaft **22**, which drives crushing rolls **2a** and **2b** as well as gripping roll **4**.

Referring to FIG. **14**, system **110** includes a conveyor **112** for introducing fibrous plant stalks to compression zone **114**. Compression zone **114** is bound by conveyor **112** and belt **116**. Belt **116** may be a cogged drive or power transmission belt and be routed over rollers **118**, **124** and **126**. Roller **118** and roller **120** partially define the paths of belt **116** and conveyor **112** respectively and act as crush rollers to further compress the plant stalks fed to system **110**. This action creates a uniform mat of fibrous plant stalk. Beyond the crush rollers, the fibrous plant stalks are directed along a feed table **122**. Feed table **122** may be positioned in a fixed manner relative to breaker **120** and scutcher **130**. In other embodiments, feed table **122** may be adjustable to allow an operator to adjust the angle at which the fibrous plant stalks are introduced to the blades **132** on breaker **128**. Blades **132** are coupled to hub **134** of breaker **128**.

Each of breaker **128** and scutcher **130** are provided with a plurality of blades for impacting the fibrous plant stalk. Scutcher **130** is provided with blades **136** coupled to hub **138**. In particular, the blades **132** and **136** may overlap such that the fibrous plant stalk is bent back and forth as it passes through system **110**. Also, the overlap of blades **132** and **134** results in a stretching action as the tips of blades **132** and **136** are raked across the fibrous plant stalk. Upon exiting the region of overlap between breaker **128** and scutcher **130**, the fibrous plant stalk is directed through a discharge oriented perpendicular to a plane that is parallel to the axis of both breaker **128** and scutcher **130**.

FIGS. **15** and **16**, depict a fibrous stalk decorticator with an adjustable feed assembly **211**, large breaker **228**, and small scutcher **230**. Breaker **228** is composed of 20 air comb blades **232** attached to hub **234**, whereas scutcher **230** is composed of 10 air comb blades **236** attached to hub **238**.

Based on this breaker and scutcher configuration the gearing drive ratio between scutcher **230** and breaker **228** must be 2:1.

Feed table **211** can be seen in two positions. In FIG. **15** it is shown flat or straight on, while in FIG. **16** it is shown at an 8 degree angle of incline. The range of adjustability will fall within the range of 0 degrees to 10 degrees. Feed table **222** is composed of spacers **223** which may be removed and inserted as determined by machine operator for the purpose of shortening or lengthening the space between the edge of feed table **222** and the orbit of breaker **228**.

Frame **210** encases the fibrous plant stalk decorticator. Baffles **248**, **250**, and **251** protect the inside and direct shattered core to the bottom of the decorticator.

Fibrous plant stalks enter the machine at input port **214**. Upon entry fibrous plant stalks are flattened over rollers **218** and **220**. Roller **218** is part of feed belt assembly **211**. Rollers **218** and **224** are fixed to the structural frame by elements suitable for applying pressure downward. Each roller **218** and **224** is capable of traveling up or down independent of one another by 0.5 inch. As fibrous plants stalks enter input port **214**, lateral baffle guide **215** prevents flattened fibrous material from laterally spilling over the edges of feed table **222**.

Feed belt assembly **211** consists of feed belt **216** which is driven by roller **226**. Feed belt **216** is a cogged timing belt constructed of polymer such as polyurethane that provides gripping action. As previously described rollers **218** and **224** provide downward pressure and are capable of traveling up and down 0.5 inch. Rollers **218** and **224** are cogged. Tensioner **240** is composed of shafts **244**, pressure device **246**, and roller **247**. The purpose of tensioner **240** is keep feed belt **216** tight.

FIG. **17** shows the concaved tips for air comb blades **252** and **254** on both the upper breaker **228** and lower scutcher **230**.

FIG. **18** is a view from the above the fibrous plant stalk decorticator depicting feed table edge **255** inversely contoured to match the concaved air comb blade tip **252**. Lateral baffle **215** is also depicted in FIG. **18**. It will be noted that air comb blade **252** exceeds the width of feed path **256**. This extra spacing on the lateral periphery of feed path helps to contain the material flow and prevent lateral spill over.

FIGS. **19-27** depict the decortication element assembly. FIGS. **19** and **20** depict inner spacer **310** of the hub assembly. The space can be built to accommodate any operational width. Space **322** is hollow and accepts air from the central shaft. Holes **323** and **324** represent air channels from the interior space **322** to the exterior of the spacer. Air combs **510** are secured to external face **314** of the spacer and external face **414** of the outer wheel hubs. Each air comb **510** is attached by a blade clamp **610** by screws through points **614**, **518**, and **416**. A gasket may be inserted on face **314** to create an air tight seal between the spacer and the air comb.

FIGS. **21** and **22** depict the outer wheel hubs **410**. Outer wheel hubs **410** are bolted on both sides of inner spacer **310** with screws through screw insertion points **326** and **426**. Inner plate faces **415** and **420** of outer wheel hub **410** fits into corresponding plate faces **312** and **320** of spacer **310**. A rubber gasket seal may be inserted between rims **318** and **418** to form an airtight seal.

Space **422** accepts the central drive shaft which is anchored by a screw inserted through hole **432**. The outer face **434** of outer wheel hub **410** also has a protruding rim lip **430** and rim face **428**. Hole **432** extends through to rim

lip **430**. A gasket or washer may be inserted around the central drive shaft and into space **422** to create an air tight seal.

FIGS. **23** and **27** depict the entire wheel hub assembly without air combs **510** and with air combs **510**. FIG. **27** represents a complete decortication element.

FIG. **24** shows the air comb **510** which is assembled by placing two L-shaped blades **528** depicted in FIG. **25** back to back. The assembled air comb is attached to the hub assembly FIG. **23** with screws through points **518**. Blade face **514** extends out from the hub assembly and comes to a rounded tip **520**. Each L-shaped blade **528** has grooves cut into the back side of blade face **514**. When the two L-shaped blades are positioned back to back these grooves form air channels running from interior bottom **526** to the ports **522** located in the rounded blade tip **520**. Sides **512** seal these channels. Lateral side gap **513** may be sealed by welding or by resin.

FIG. **26** depicts the blade clamps which secure air combs **510** to the wheel hub assembly. Angles **612** correspond to angles **316** FIGS. **19** and **20** and angles **416** FIGS. **21-22**. Blade clamp **610** fits over the edges of air comb **510**. Screws inserted through holes **614** correspond to holes **518** in air comb **510** and holes **424** in outer wheel hub **410**. In addition, the method of decortication described in this disclosure provides a suitable elements for constructing a highly efficient and effective decortication machine capable of achieving the rapid throughput capacities required to economically operate said decortication machine in cooperation with modern forage or combine harvesting equipment, as well as equal or exceed the throughput of modern stationary decortication systems.

There are undoubtedly numerous modifications and arrangement of parts that may be construed from this invention. The previous description provides guidelines for building and operating this invention and is not meant to limit the operation of this invention, in so far as potential modifications and arrangements relate to the following claims.

Consistent with the description herein and the appended drawings, some features of exemplary embodiments include:

- 1) A variable throughput decortication machine possessing a unique decortication element, which effectively and efficiently isolates the outer bast fiber of plant stalks from the inner woody core thereof, comprising the following arrangement of working parts:
 - a. Feeding elements consisting of a lower and upper feed belt assembly that compresses plant stalks into a uniform mat and conveys said uniform mat over a feed table and under the influence of a gripping feed roll. It is the intention, ability, design, and purpose to achieve such result that is claimed novel.
 - b. A feed table with an adjustable angle of inclination and length composed of metal or suitable composite material and coated with a slick hydrophobic coating or finish for reducing friction and improving flow ability as the uniform mat is driven over the feed table and under the influence of the gripping feed roll.
 - c. An upper feed belt composed of a gripping polyurethane, such as "sure grip," to securely hold the fibrous mat while simultaneously feeding it through the decortication element so that the entire length of the plant stalks contained in the mat are decorticated.

11

- d. A lateral baffle or guide assembly between the crushing rolls and through the gripping feed roll that contains the uniform mat and prevents it from lateral escape.
- e. A feed table edge that is inversely contoured to match the concave tip of the breaker air comb blade tips.
- i. The distance of clearance between the orbital circumference of the upper breaking roll and the edge of the feed table is adjustable within a range of 5 millimeters and 44 millimeters. This range of distance provides for optimal fiber cleaning, minimizes damage, and allows for operation on fibrous plant stalks of varying degrees of moisture content.
- j. Breaking and scutching rolls composed of 2 outer hubs and a hollow center spacer.
- k. The assembly is either circular or polygonal in shape and attaches to hollow central drive shaft capable of delivering air to the central spacer which is hollow. Air comb blades attach to the outer circumference of the hubs. The center spacer is hollow and has channels cut to the outer circumference that deliver air to the air combs.
- l. Air comb blades are made of metal or other suitable hardened material capable of withstanding rapid repeated impacts with fibrous materials of wet or dry physical state and capable of delivering forced air in any form: jet, stream, band, etc. in relative proximity to the point of contact with fibrous materials and from the extremity or tip of the air comb blade. It is understood that there are numerous ways for achieving this result. It is the intention, ability, design, and purpose to achieve such result that is claimed novel.
- m. The orbital paths of the breaking and scutching rolls intermesh within the circumferential paths of the air comb blades. The degree or severity of intermeshing may be controlled by adjusting the height of the air comb blades on either the scutching or breaking rolls.
- n. A novel decortication element design consisting of an upper breaking roll that has a greater outer diameter than the lower scutching roll. Said assembly creates a uniquely beneficial sweeping action during the intermeshing of the orbital paths of the breaking and scutching rolls. Specifically, the lower scutching roll travels at a faster peripheral speed than the upper breaking roll. As a result, when the lower scutcher blade enters the orbital path of the upper breaking roll, a point in space referred to as the pitch point, it will gain distance from the trailing breaking roll blade and gain distance on the leading breaking roll blade throughout the period of intermeshing orbits. This action creates an additional bending and flexing of the fibrous material which advantageously loosens and frees core from the fiber. It will be noted that by altering the pitch point of the lower scutching roll blades' orbit in relation to the breaking roll blades' orbit that a variety of sweeping effects may be provided. Other novel aspects are not limited to this design innovation and may be incorporated in a breaking and scutching roll assembly that is composed of two rolls of equal outer diameter and number of blades.
1. Air comb blades and feed table edge are rounded and smoothed in such a manner that there are no sharp edges that could sever or cut the fibrous material during decortication. These edges may either be smooth or cut with longitudinally aligned micro-

12

- combing grooves running parallel to the path of travel of the fibrous materials. The rounded and smooth edges minimize fiber damage by reducing friction and allowing fibrous materials to slip across the surface. The rounded edges with micro-combing grooves have similar benefits but also enhance the ability to open the fiber bundles during decortication which is beneficial.
- 2) A novel air comb design for breaking and scutching roll blades that functions to reduce fiber damage, improve fiber cleaning and opening, enhance material handling, and simplify construction and repair, consisting of:
- a. Two L-shaped pieces of suitable metal with concaved tips and bases that attach to breaking and scutching hub and spacer assembly, otherwise referred to as rims, provide a unique blade assembly for the decortication element. The two L-shaped pieces are symmetrical and fit together back to back. They may be sealed airtight by welding together, resin, or any other suitable ways.
- b. The outer surface of the air comb blade may be finished or coated with a slick hydrophobic coating or finish for reducing friction and improving flow ability.
- c. Air comb blades attach to the exterior rims and extend longitudinally across the surface.
- d. Air comb blades may be angled directly perpendicular to the directional flow of the fibrous material or at an offset angle forward or backward.
- e. Pluralities of channels are cut into the back side of the individual L-shaped blades that reduce in volume from the base to the tip of the individual blade. These channels form jet nozzles when the individual blades are joined together back to back.
- f. The base of the air comb blades have male key grooves and male ports that correspond to female key grooves and female ports that are cut into the exterior of the breaking and scutching rolls. Male-female key grooves and ports allow for simple fastening and detachment, as well as easy angular adjustment, forward and backward, with respect to direction of orbital travel by way of screws or similar control and fastening mechanism.
- 3) A novel breaking and scutching roll assembly consisting of 2 outer hubs and 1 inner spacer.
- a. The inner spacer may be constructed to accommodate any lateral width.
- b. The outer hubs may be journal and keyed in to lock the air comb blades in place.
- c. The inner spacer is hollow to accept air from the central shaft and may include an internal baffle to direct air flow to a limited section of the inner circumferentially arc.
- d. The assembly may be sealed air tight by a variety of ways including but not limited to gaskets, washers, and resins.
- 4) A decortication machine according to claims 1, 2, and 3 wherein the feed element and decortication elements are controlled by separate variable speed drives that allow the number of impacts per forward travel to be precision calibrated.
- 5) A plant stalks decortication method that isolates the outer bast fiber from the woody core thereof, consisting of the following beneficial actions.
- a. An initial compression and crushing action between an lower and upper belt and pressure rolls serves to break down the bonds between the bast fiber skin and

13

- woody interior of the fibrous plant stalks, creating a uniform mat of flattened fibrous plant stalks for decortication, removes moisture and gums, and drives the mat forward over the feed table edge and into the orbit of the decortication elements. 5
- b. A second gripping feed roll that is part of the upper feed belt assembly grips the fibrous mat as it is fed forward and acted up by the decortication elements. The upper belt should be composed of a polymer such as polyurethane that is capable of gripping the material and not allowing it to slip forward. 10
- c. A third breaking action bends and flexes the fibrous material down over the feed table between 60 to 120 degree angle shattering the woody core and loosening the bast fiber from the core. 15
- d. A fourth scutching action catches the advancing fibrous material in substantially the same place as the previous breaking impact bending the advancing fibrous material back the opposite direction at a 60 to 120 degree angle and forcing it to momentarily be subjected to additional bending and flexing as it is caught between the intermeshing orbits of the breaking and scutching roll blades orbital paths. The additional bending and flexing causes the shattered and loosened core to be beaten or whipped from the fiber. 20 25
- e. A fifth combing, opening, and cleaning action from combing grooves, micro or otherwise, that are cut into the feed table and tips of the air comb blades on both the breaking and scutching rolls and run parallel to the directional flow of the fibrous material that serve to open the fiber bundles and improve fiber cleaning. 30
- f. A sixth pneumatic cleaning action comprised of a forceful air blast delivered from the peripheral extremity of the breaking and scutching roll blades, such device otherwise referred to as air comb blades, and delivering said air blast at the point of contact with the fibrous materials thereby effectively removing core from the fibrous materials, opening the fiber bundles, reducing moisture, preventing fiber from wrapping around moving parts, and aiding in post-decortication fiber recovery. 35 40
- g. A seventh pneumatic action consisting of blasting chilled air through the pneumatic cleaning element, i.e., air comb, for the purpose of causing the woody core to more rapidly and easily separate from the fibrous materials. 45
- h. An eighth pneumatic action inversely consisting of blasting heated air through the pneumatic cleaning element, i.e., air comb, for the purpose of rapidly reducing the moisture content of the fibrous materials. 50
- i. A ninth decortication action derived from engineering the lower scutching roll to travel at greater peripheral speed than the upper breaking roll to increase the cleaning efficiency of said lower scutching roll. 55
- j. A tenth friction reducing action provided by the slick hydrophobic coating, finishing, or material applied to the feed table, feed table edge, and all air comb blade tips that allows the fibrous materials to slip and flow through and over the previously listed elements. 60
- k. An eleventh lateral guiding action throughout the process beginning in the feed element with lateral baffles and continuing through the breaking and scutching rolls with concave blade tips. Lateral control is also enhanced by a decortication element 65

14

width that exceeds the feed element width by 0.5 or more inches on either side.

- 6) A plant stalk decortication method according to claim 5 that utilizes an air blasting feature located in the peripheral tips of the breaking and scutching blades, and otherwise referred to as an air comb, to minimize the duration and degree of contact between said blades and the fibrous materials, resulting in less severe bending and depth of intermeshing between the blades of the breaking and scutching roll air comb blades, which helps to minimize mechanical fiber damage while enhancing material handling post-decortication.
- 7) A plant stalks decortication method according to claim 6 that permits variable throughput capacities based on the operational width of the breaking and scutching rolls and that may be mounted on a harvest head implement, pick-up head implement, or utilized as part of a stationary mill.

What is claimed is:

1. A fibrous plant stalk decorticator comprising:
 - a feeding device;
 - a breaker having a first plurality of blades;
 - a scutcher having a second plurality of blades, the scutcher positioned such that the first and second pluralities of blades overlap; and
 - a plurality of air outlets passing through one or more blades and configured to provided pressurized air to a fibrous plant stalk as it is passed through the decorticator.
2. A decortication element for use in a fibrous plant stalk decorticator, the decortication element comprising:
 - a wheel hub assembly comprising:
 - an inner face, wherein the inner face is configured for attachment to a drive shaft; wherein the inner face comprises an interior air inlet for receiving a flow of air and directing the flow of air through the interior of the wheel hub assembly;
 - an outer face, wherein the outer face comprises an air outlet for receiving a flow of air from the interior of the wheel hub assembly and directing the flow of air out from the wheel hub assembly; and
 - one or more blades, wherein at least one blade is comprised of a blade air channel extending through the center of the blade and out through a top edge portion of the blade, wherein the blade air channel is configured to receive air from the wheel hub assembly and to direct it to a fibrous plant stalk as it is passed through the decorticator.
3. *The system of claim 1, wherein the system further comprises:*
 - a feeding assembly, wherein the feeding assembly comprises an upper feed belt assembly and a lower feed belt assembly; wherein the upper feed belt assembly comprises an upper crushing roll; wherein the lower feed belt assembly comprises a lower crushing roll; wherein the upper and lower crushing rolls are vertically aligned and spaced to form a first intake gap; wherein the upper and lower crushing rolls are configured to rotate in opposite directions;*
 - wherein the upper and lower crushing rolls are configured to compress plant material inserted into the first intake gap and to expel the plant material in the form of a fibrous mat of compressed material;*
 - a feed table, wherein the feed table comprises a top surface and a feed table edge;*
 - a gripping feed roll, wherein the gripping feed roll is configured to frictionally engage with the fibrous mat*

15

as it is expelled from the first intake gap; wherein the gripping feed roll is configured to propel the fibrous mat over the feed table edge; wherein the gripping feed roll is configured to rotate in the same direction as upper crushing roll;

an upper breaking roll, wherein the upper breaking roll comprises a first plurality of blades extending from an outer hub; wherein the upper breaking roll is mechanically engaged with an upper shaft; wherein the upper shaft is configured to rotate the upper breaking roll at a first speed in the same direction as the upper crushing roll;

a lower scutching roll, wherein the lower scutching roll comprises a second plurality of blades extending from an outer hub; wherein the lower scutching roll is mechanically engaged with a lower shaft; wherein the lower shaft is configured to rotate the lower scutching roll at a second speed in a second direction; wherein the second speed is faster than the first speed; and

a guide assembly, wherein the guide assembly extends between the upper crushing roll and the feed table edge; further wherein the guide assembly is positioned and configured to prevent lateral movement of the stalks past the ends of the upper crushing roll and the feed table;

wherein the orbital paths of the first and second plurality of blades intermesh during rotation;

wherein at least one of the air outlets comprises a blade air channel within at least a first surface of the at least one of the first or second plurality of blades.

16

4. The system of claim 3, wherein the feed table edge comprises a rounded edge with micro-combing grooves.

5. The system of claim 3, wherein the width of the scutching and breaking rolls exceeds the width of the feed element by more than 0.5 inches on either side.

6. The system of claim 5, wherein the upper breaking roll is configured to bend the fibrous mat down at a first location over the feed table edge in a first direction to an angle of between 60-120 degrees.

7. The system of claim 6, wherein the lower scutching roll is positioned so the fibrous mat is bent at a second location as the outer edge of a scutcher blade intermeshes into the space between the first breaker blade and the feed table edge.

8. The system of claim 7, wherein the feed table has an adjustable angle of inclination.

9. The system of claim 8, wherein the feed table edge is inversely contoured.

10. The system of claim 9, wherein the distance of clearance between the orbital circumference of the upper breaking roll and the edge of the feed table is adjustable within a range of 5-44 millimeters.

11. The system of claim 3, wherein the feed table comprises a conveyor.

12. The system of claim 1, wherein at least one of the air outlets comprises a blade air channel within at least a first surface of the at least one of the first or second plurality of blades.

* * * * *