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**Kah, Jr.**

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(54) **CENTRIFUGAL DIRT SEPARATION CONFIGURATIONS FOR HOUSEHOLD-TYPE AND SHOP-TYPE VACUUM CLEANERS**

(58) **Field of Classification Search**  
CPC ..... A47L 5/14; A47L 9/0477; A47L 9/1608;  
A47L 9/1641; A47L 9/1683; B01D 45/16;  
(Continued)

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(56) **References Cited**

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(21) Appl. No.: **15/842,289**

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55/345  
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(22) Filed: **Dec. 14, 2017**

(Continued)

**Related U.S. Patent Documents**

**OTHER PUBLICATIONS**

Reissue of:

International Search Report, Jun. 11, 2012.

(64) Patent No.: **9,119,511**  
Issued: **Sep. 1, 2015**  
Appl. No.: **13/209,867**  
Filed: **Aug. 15, 2011**

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U.S. Applications:

(57) **ABSTRACT**

(60) Division of application No. 15/147,518, filed on May 5, 2016, which is an application for the reissue of Pat. No. 9,119,511, which is a continuation of application No. 12/074,438, filed on Mar. 3, 2008, now Pat. No. 7,996,957.

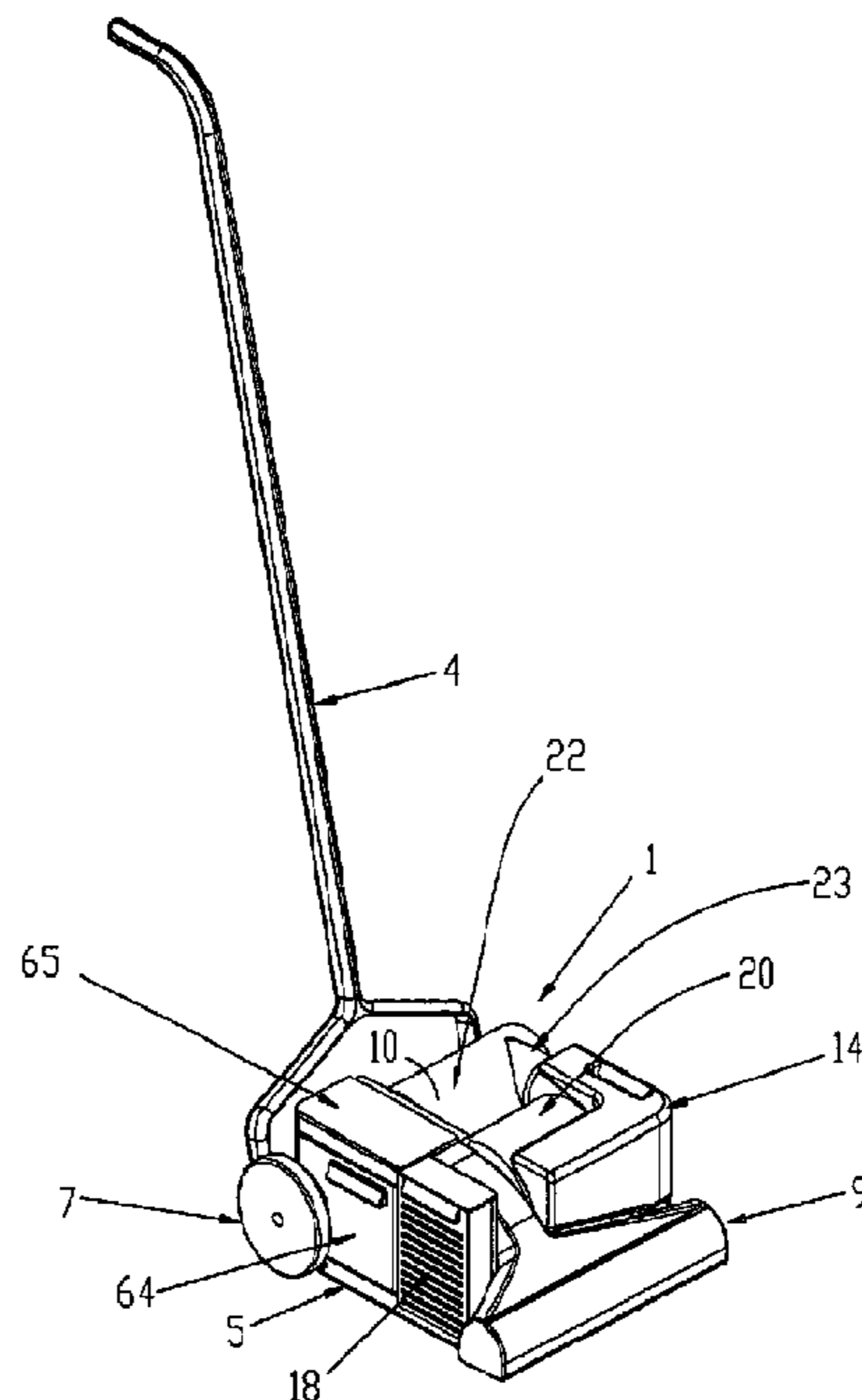
A cyclonic separation device in accordance with an embodiment of the present application preferably includes a first cyclone chamber having a cylindrical shape with a predetermined diameter, the first cyclone chamber including, a tangential inlet positioned on a first longitudinal end of the first cyclone chamber, a baffle plate positioned in the first cyclone chamber a predetermined distance from the tangential inlet, a tangential dirt outlet positioned on a second end of the cyclone chamber, opposite the inlet and on an opposite side of the baffle plate from the tangential inlet and a center exit duct mounted in the center of the cyclone chamber having an inlet opening positioned upstream from the baffle plate such the centrifuged fluid without particles flows into the center exit duct and out of the cyclone chamber.

(Continued)

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**B01D 45/16** (2006.01)  
(Continued)

(52) **U.S. Cl.**  
CPC ..... **A47L 5/14** (2013.01); **A47L 9/0477** (2013.01); **A47L 9/1608** (2013.01);  
(Continued)

**5 Claims, 28 Drawing Sheets**





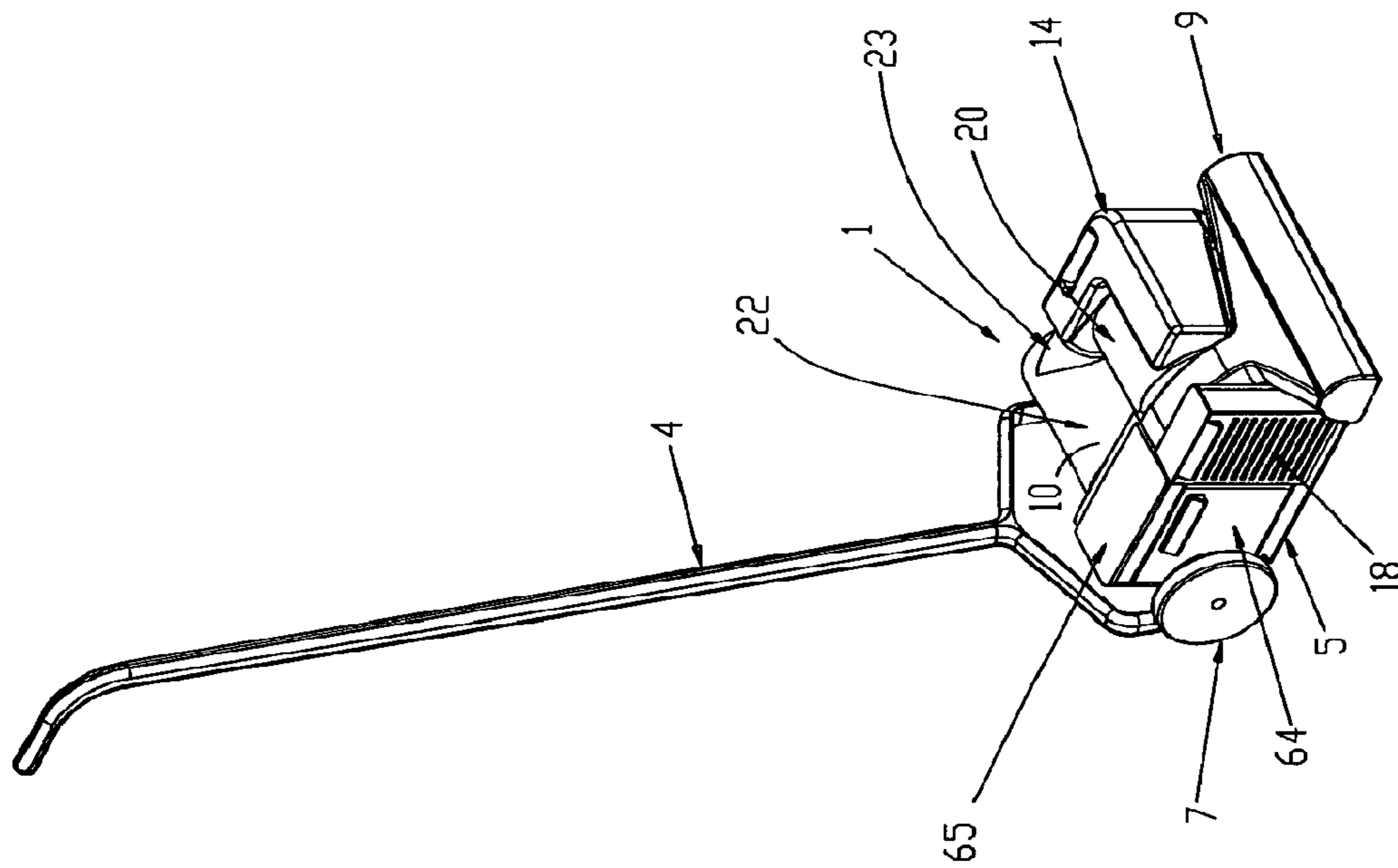


Figure 1

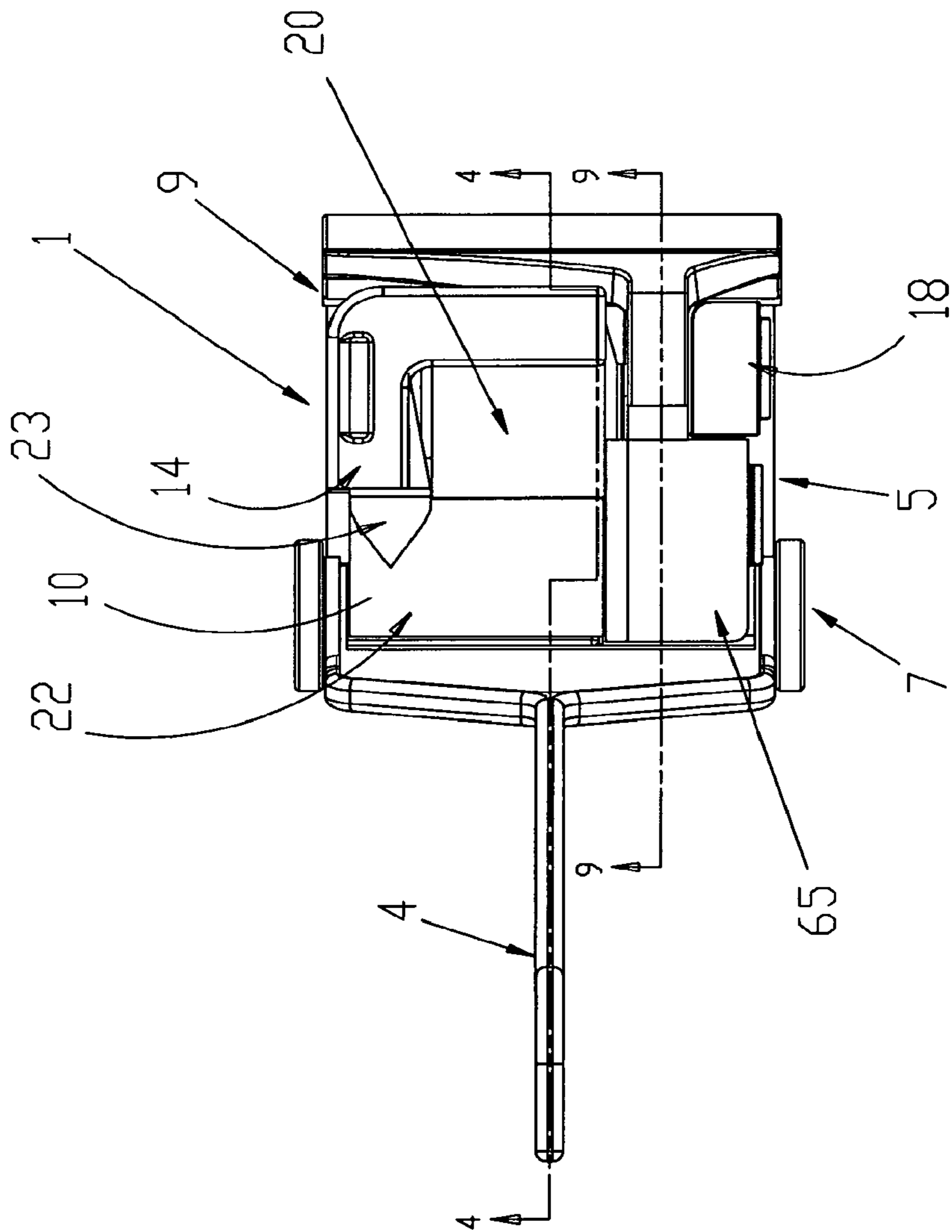


Figure 2

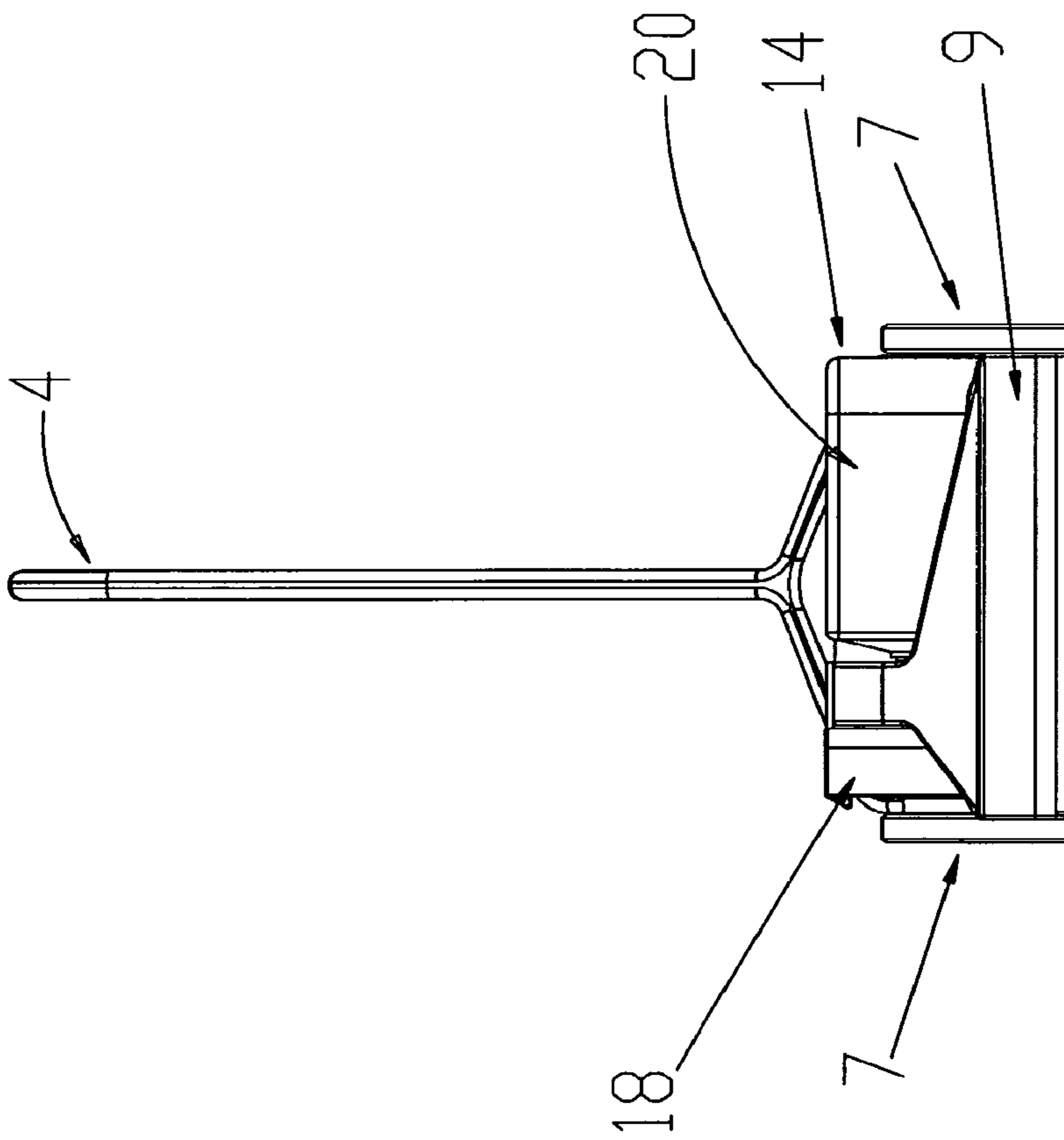


Figure 3



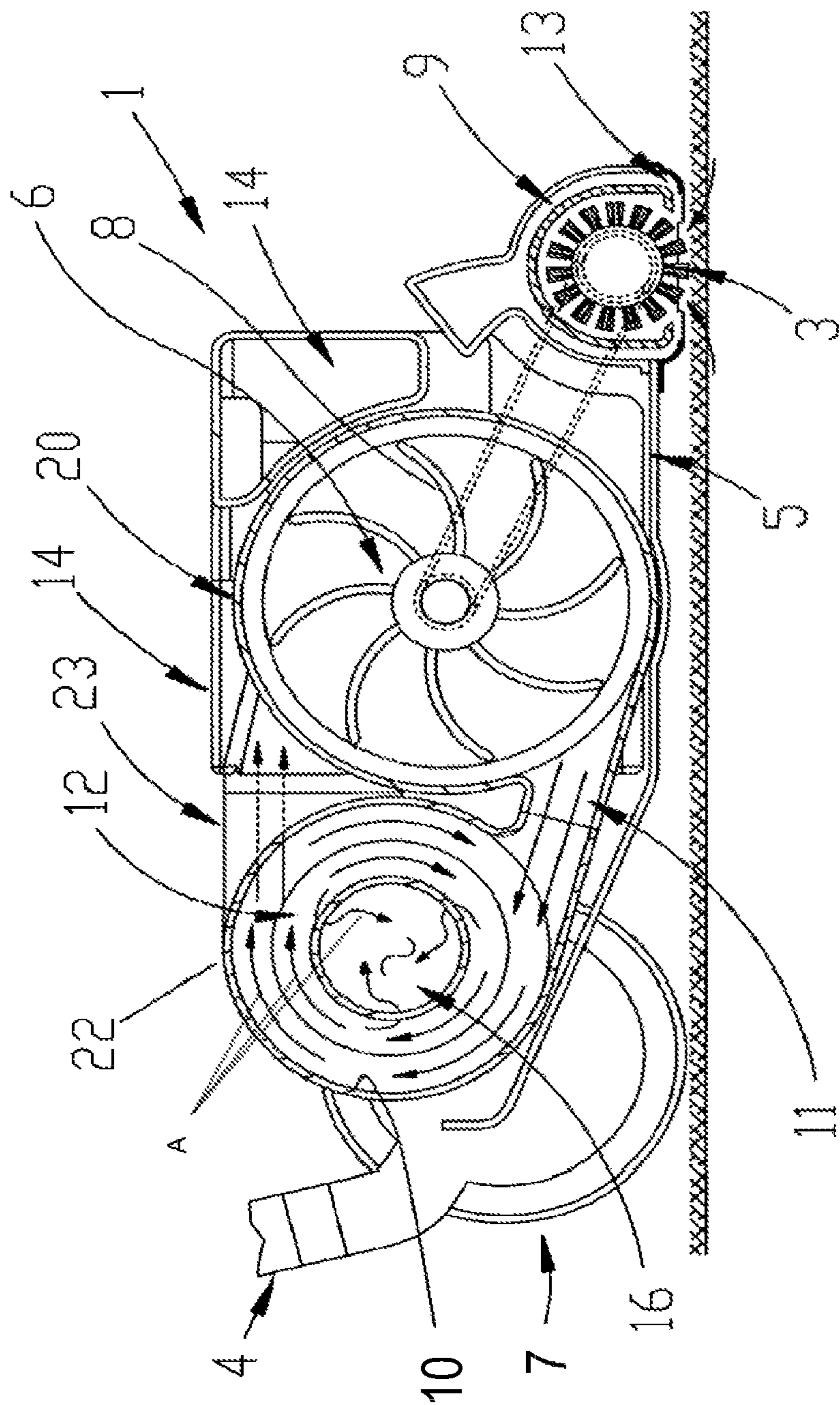


Figure 4

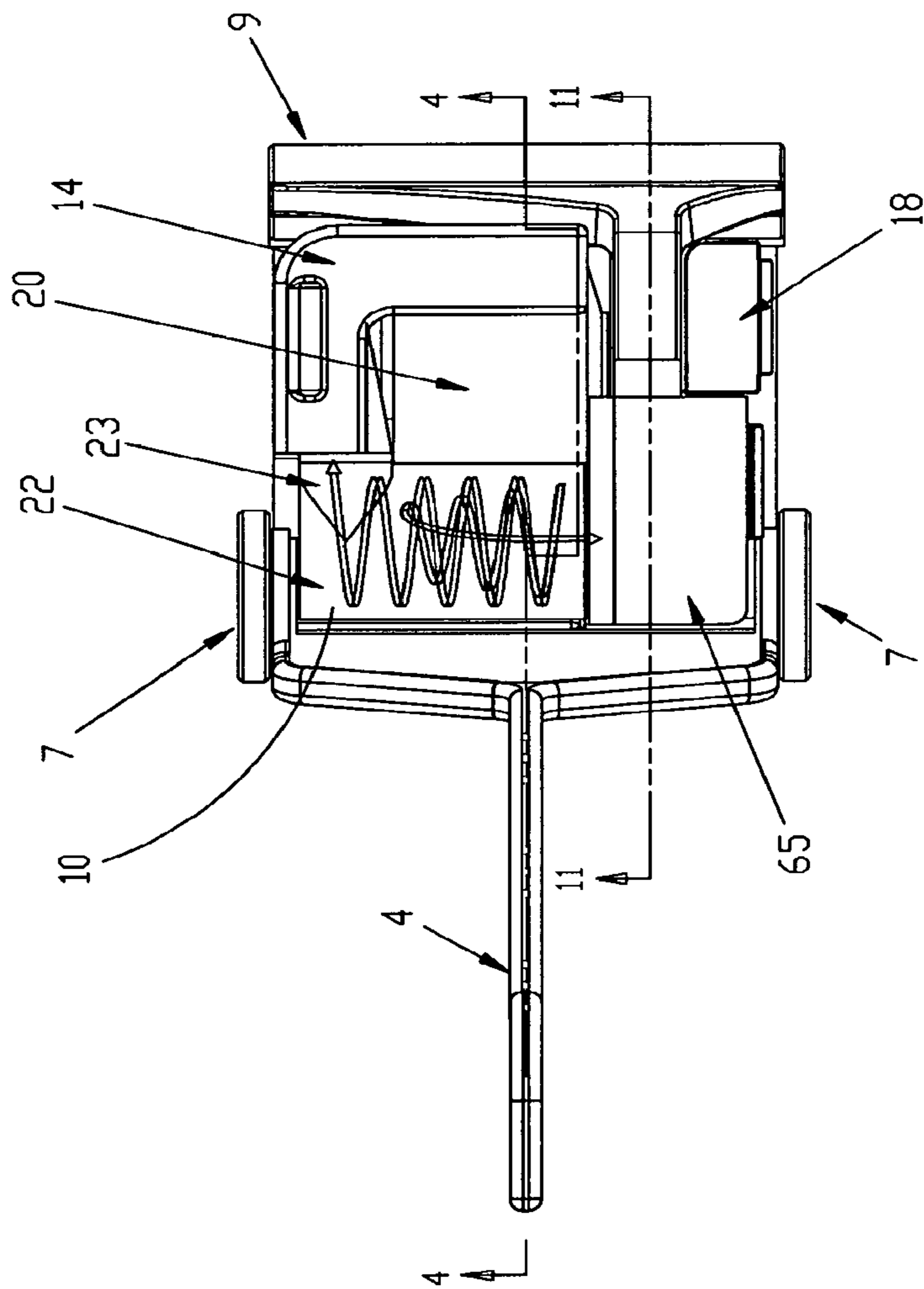


Figure 5

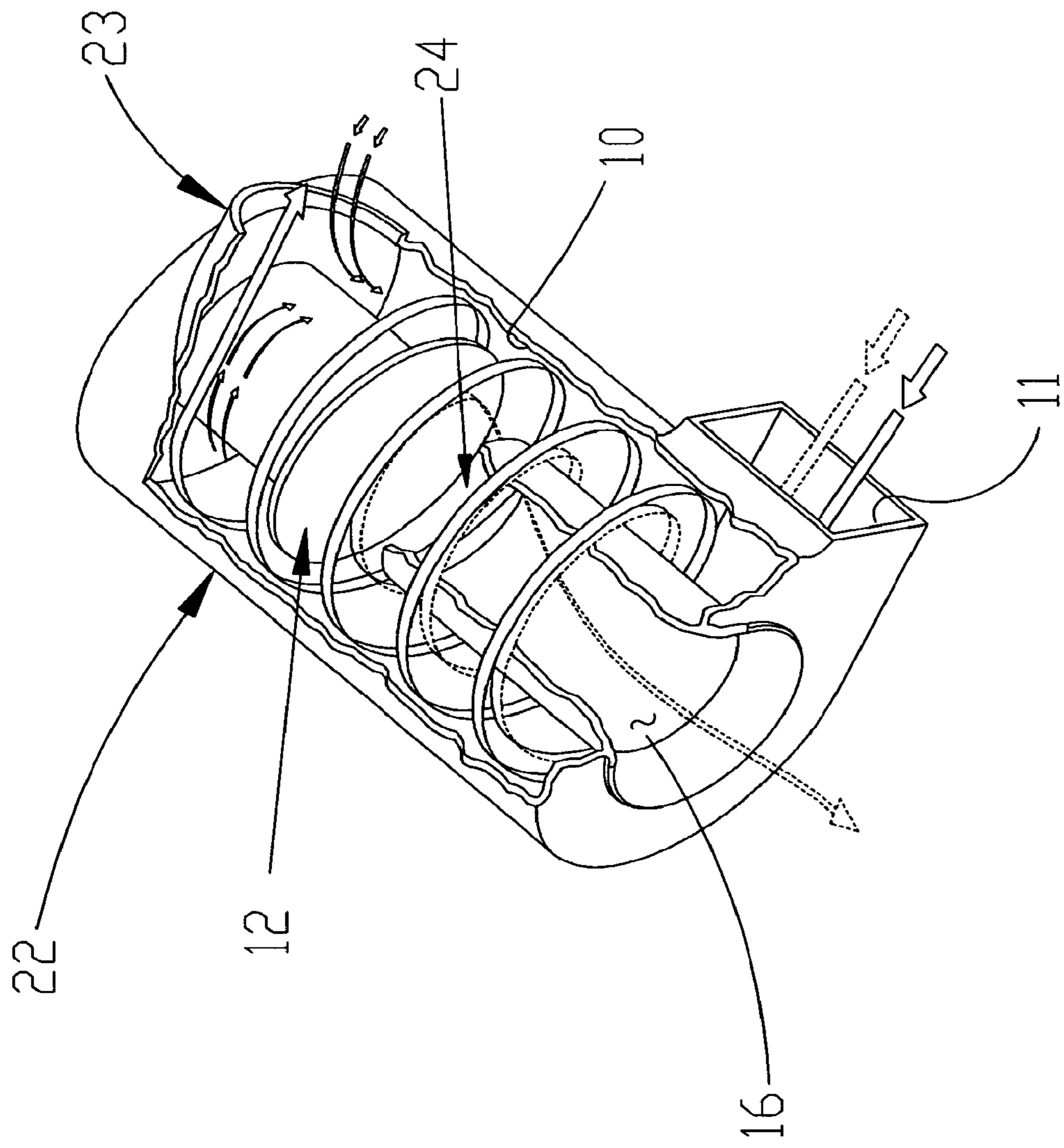


Figure 6



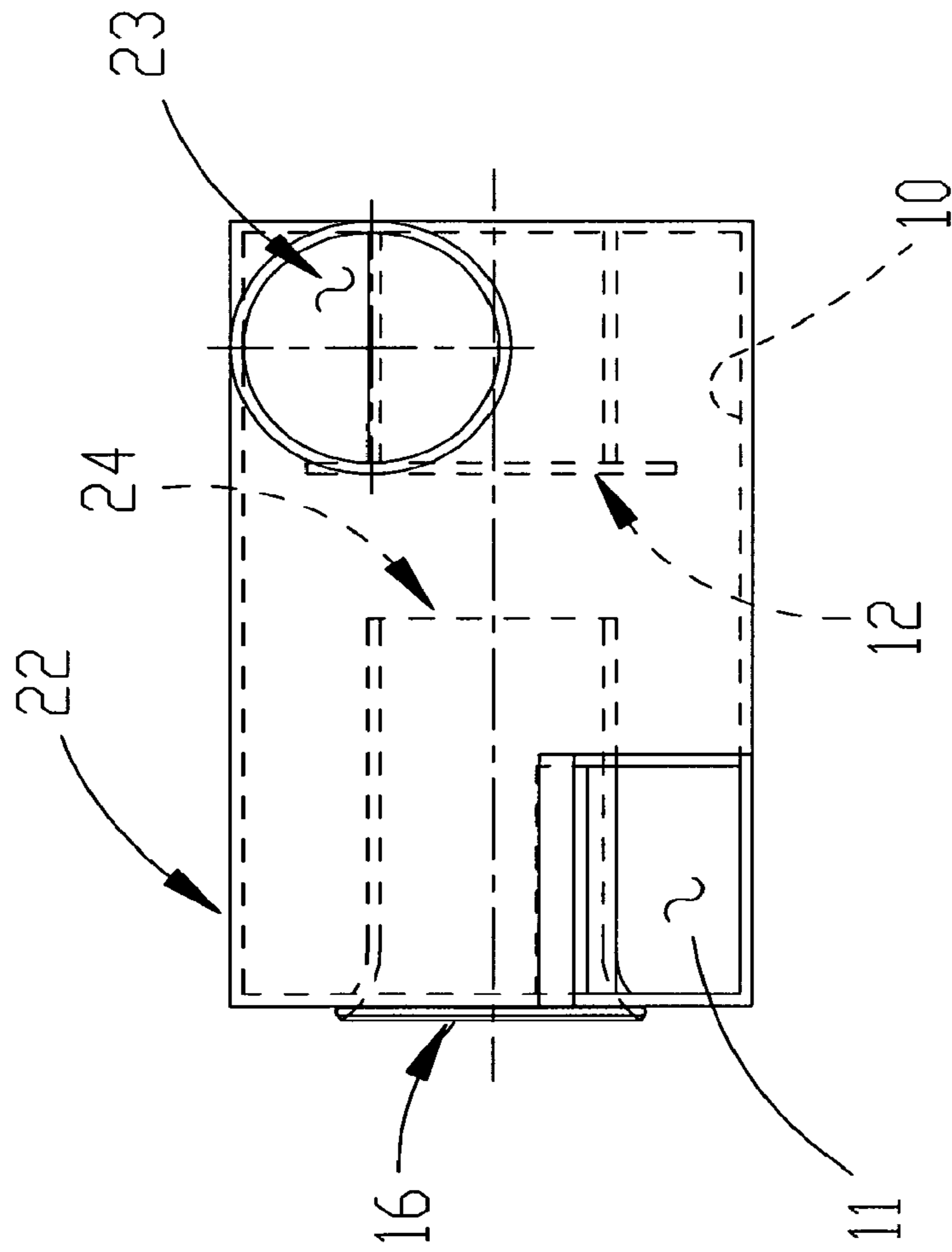


Figure 7

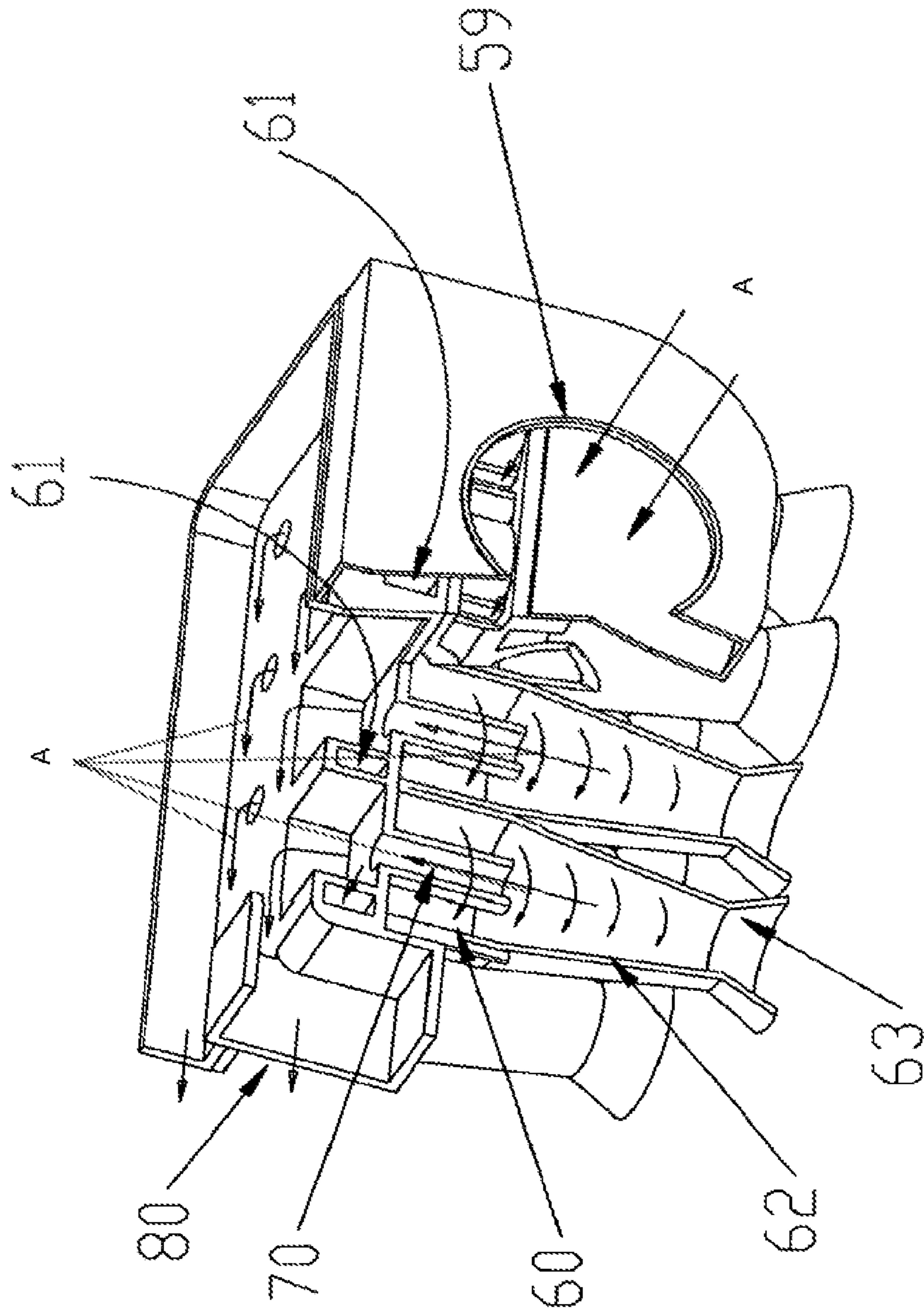
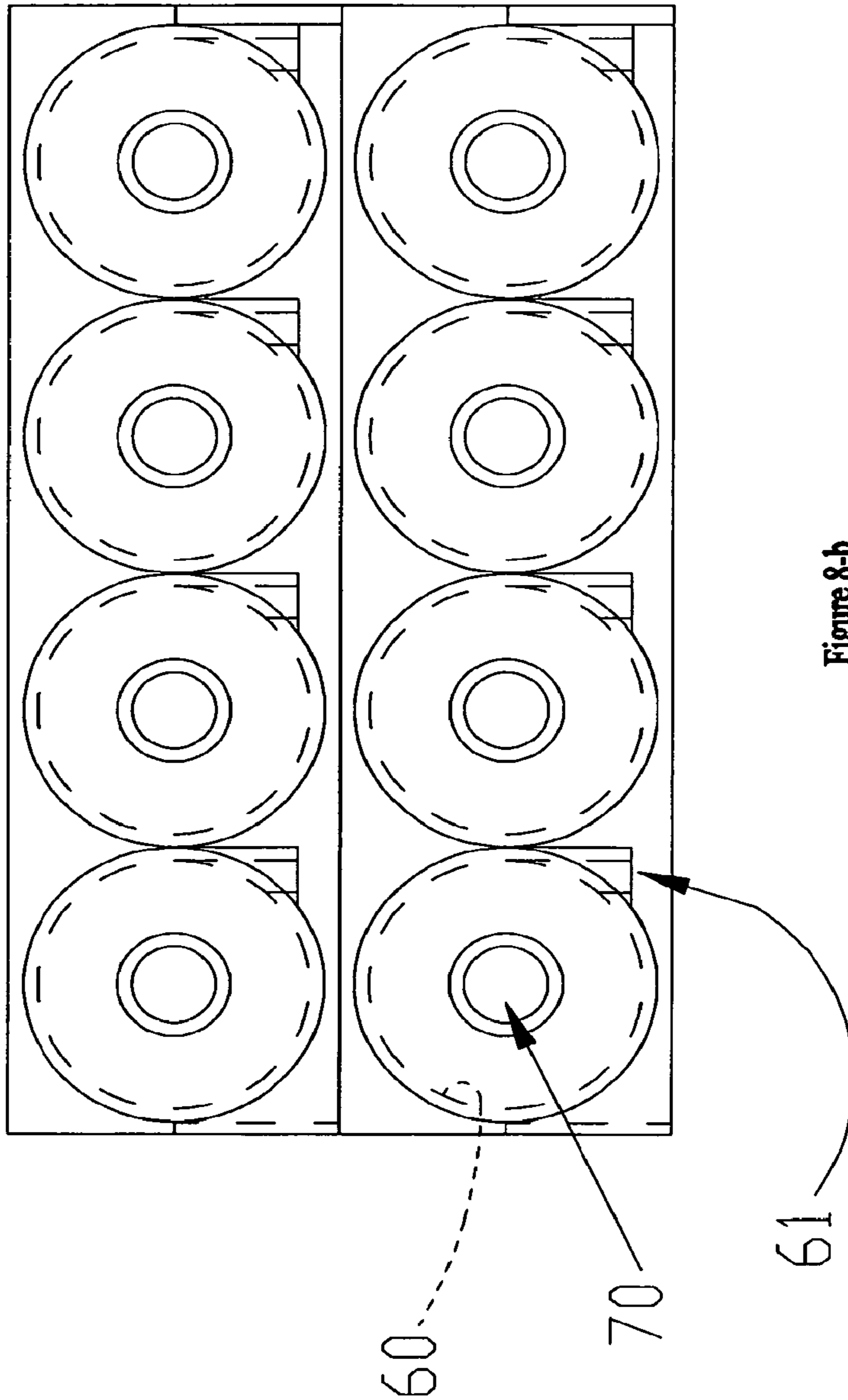


Figure 8a



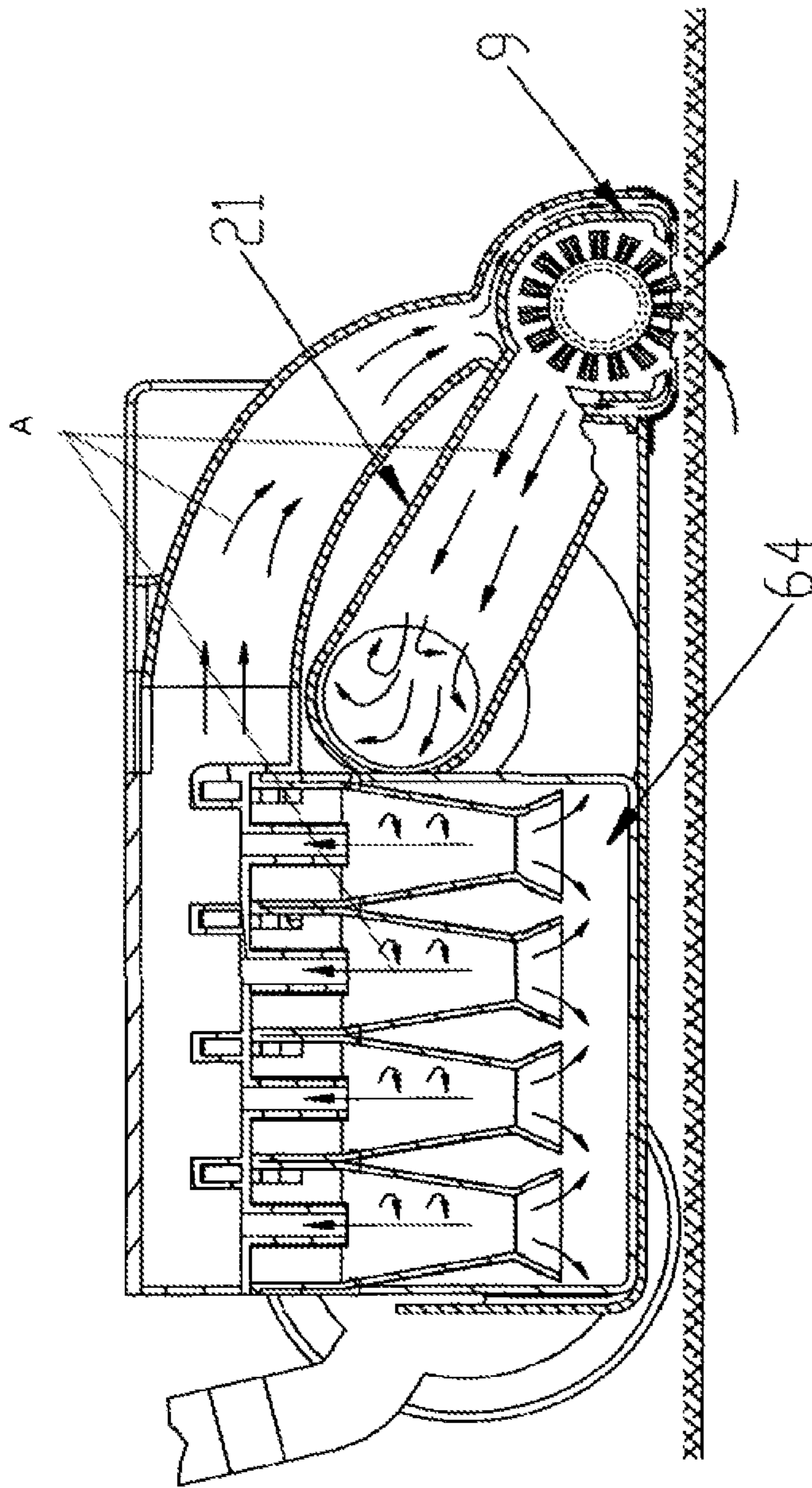


Figure 9

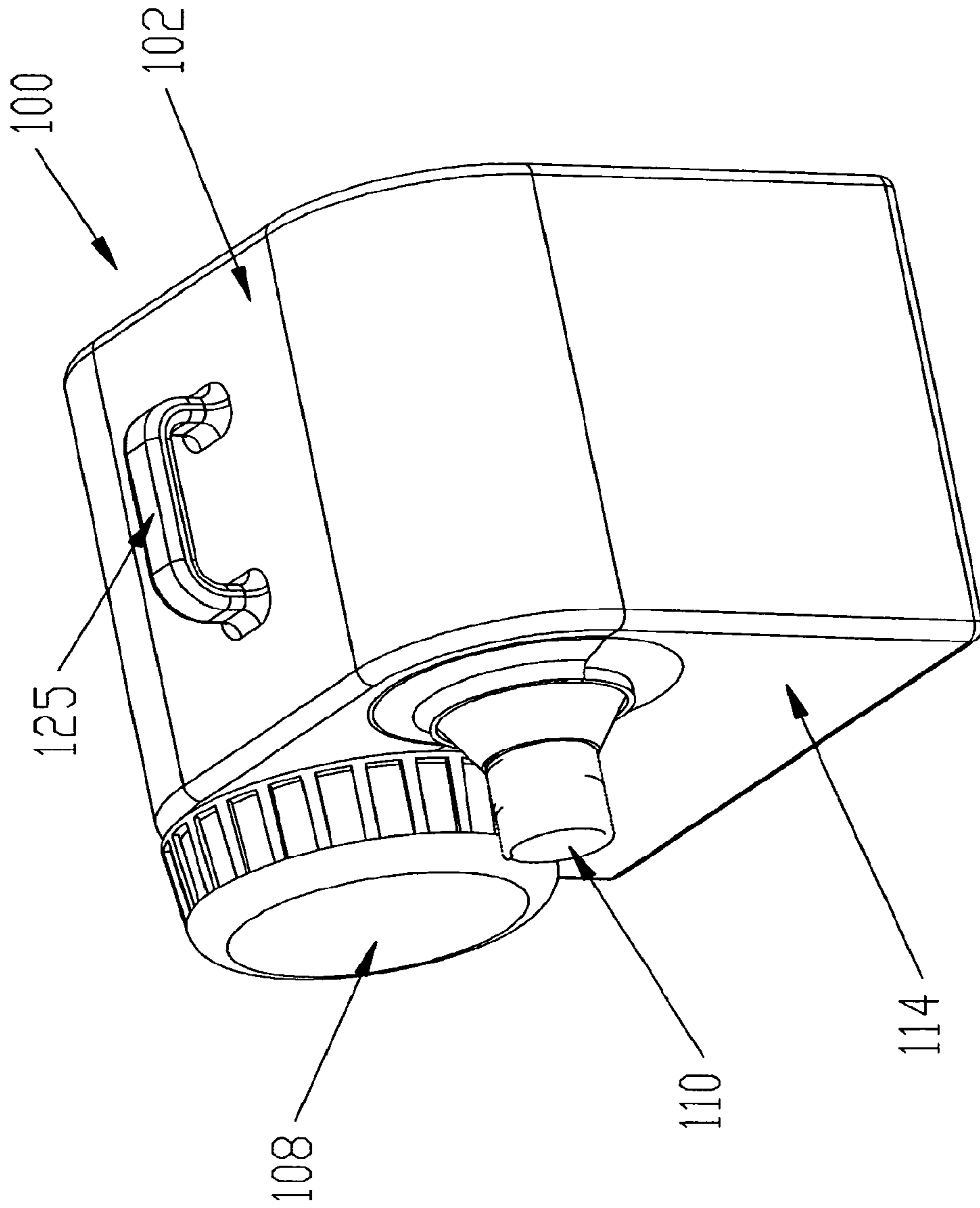


Figure 10a



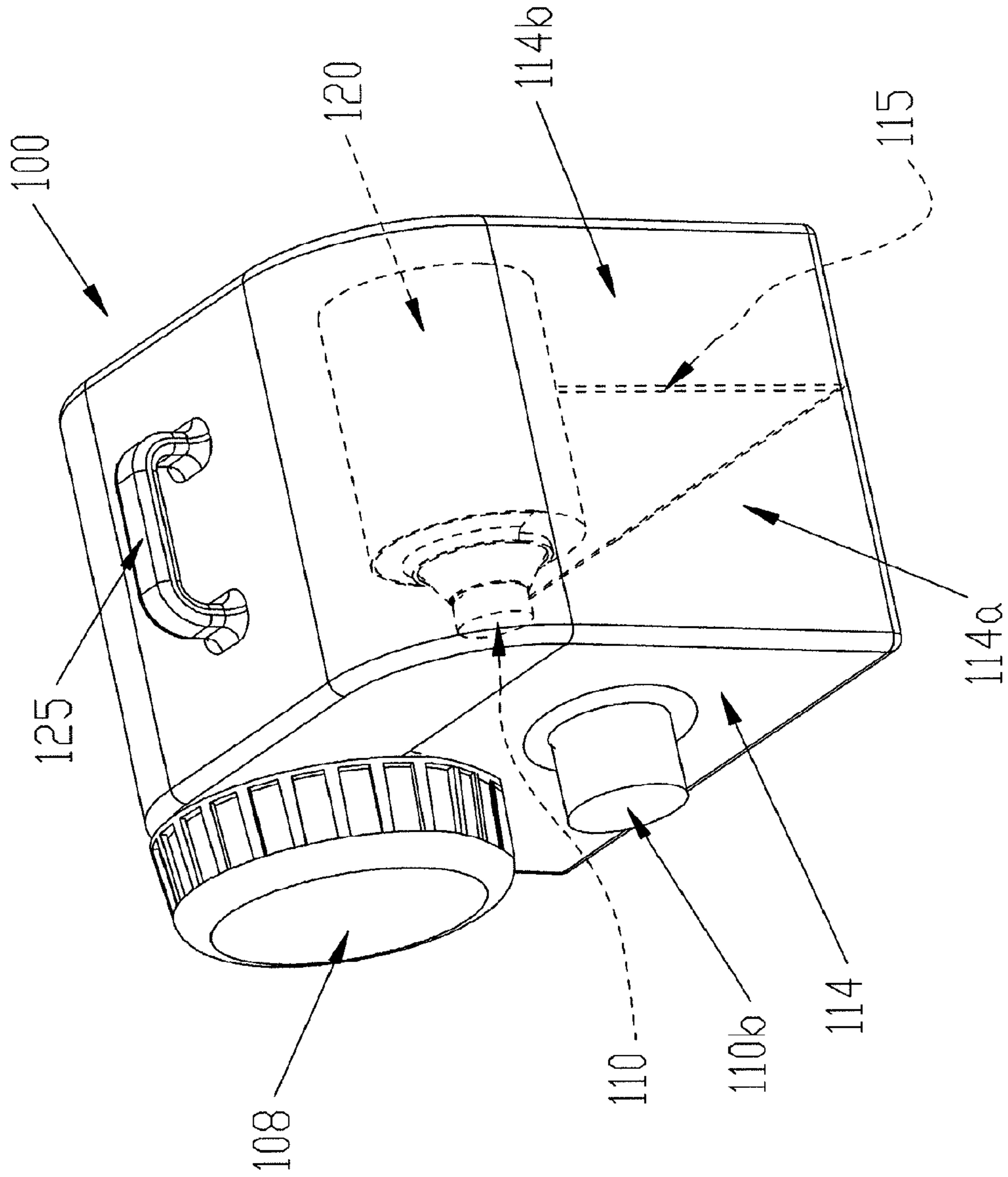


Figure 10b

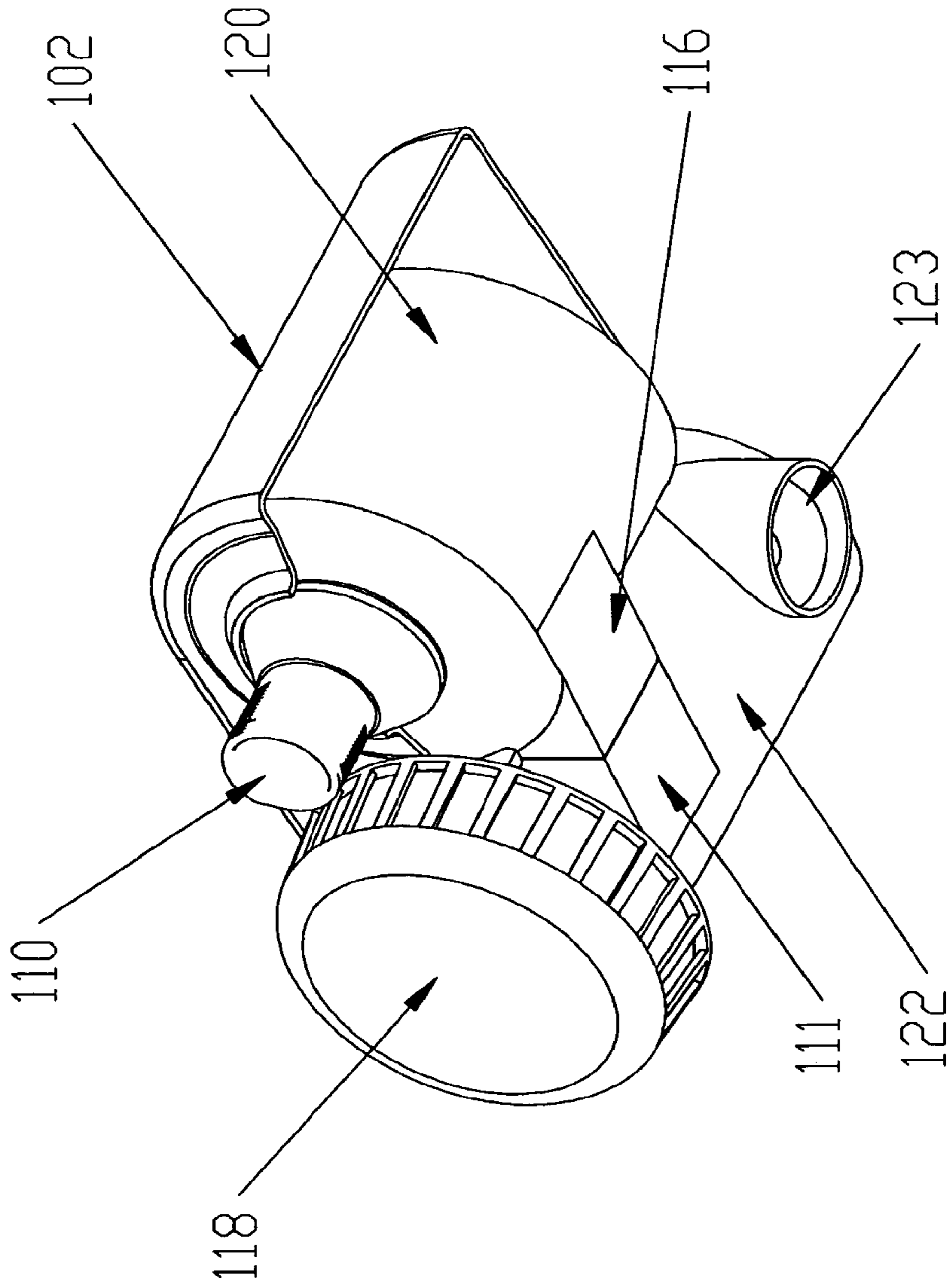


Figure 11a

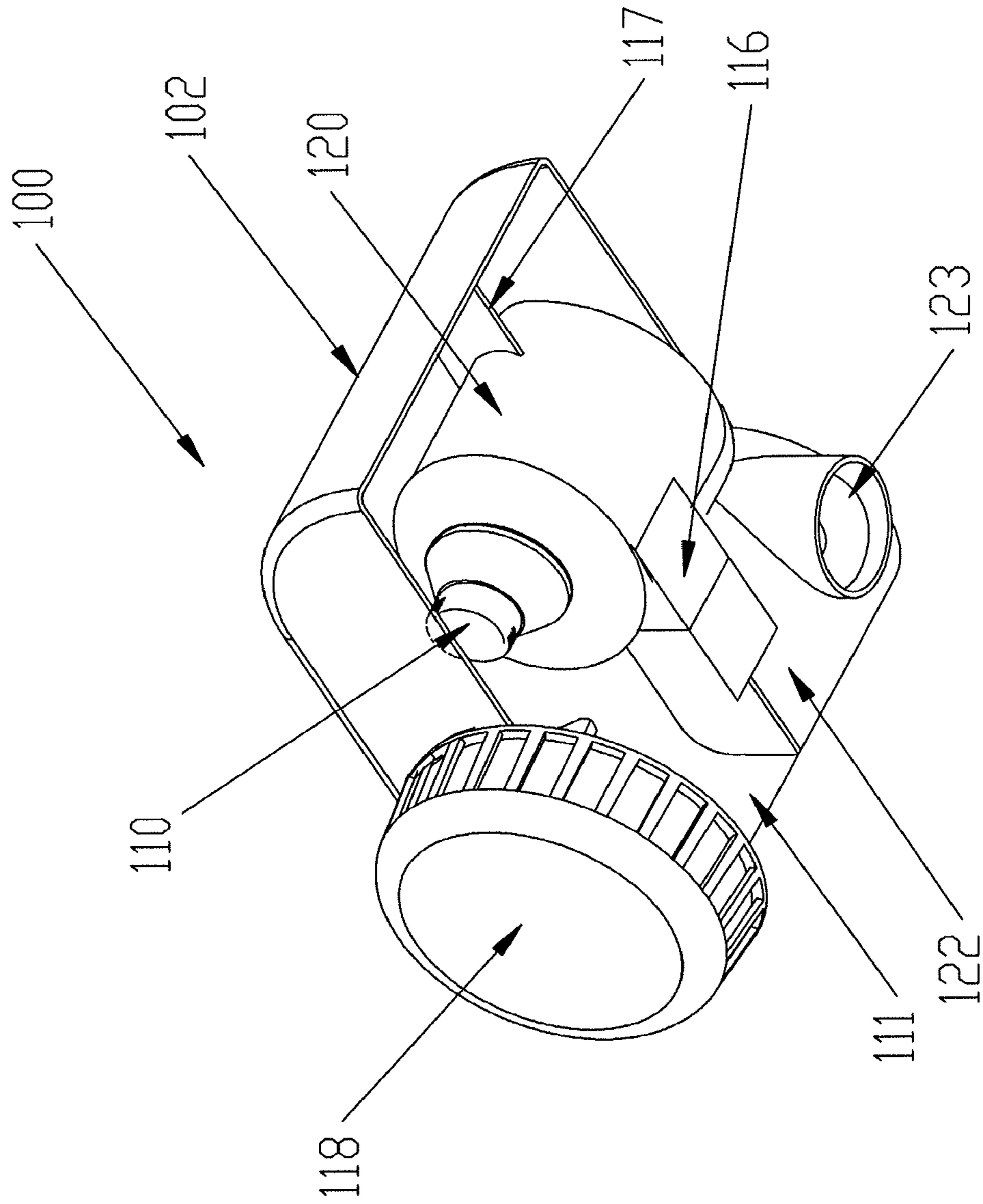


Figure 11b

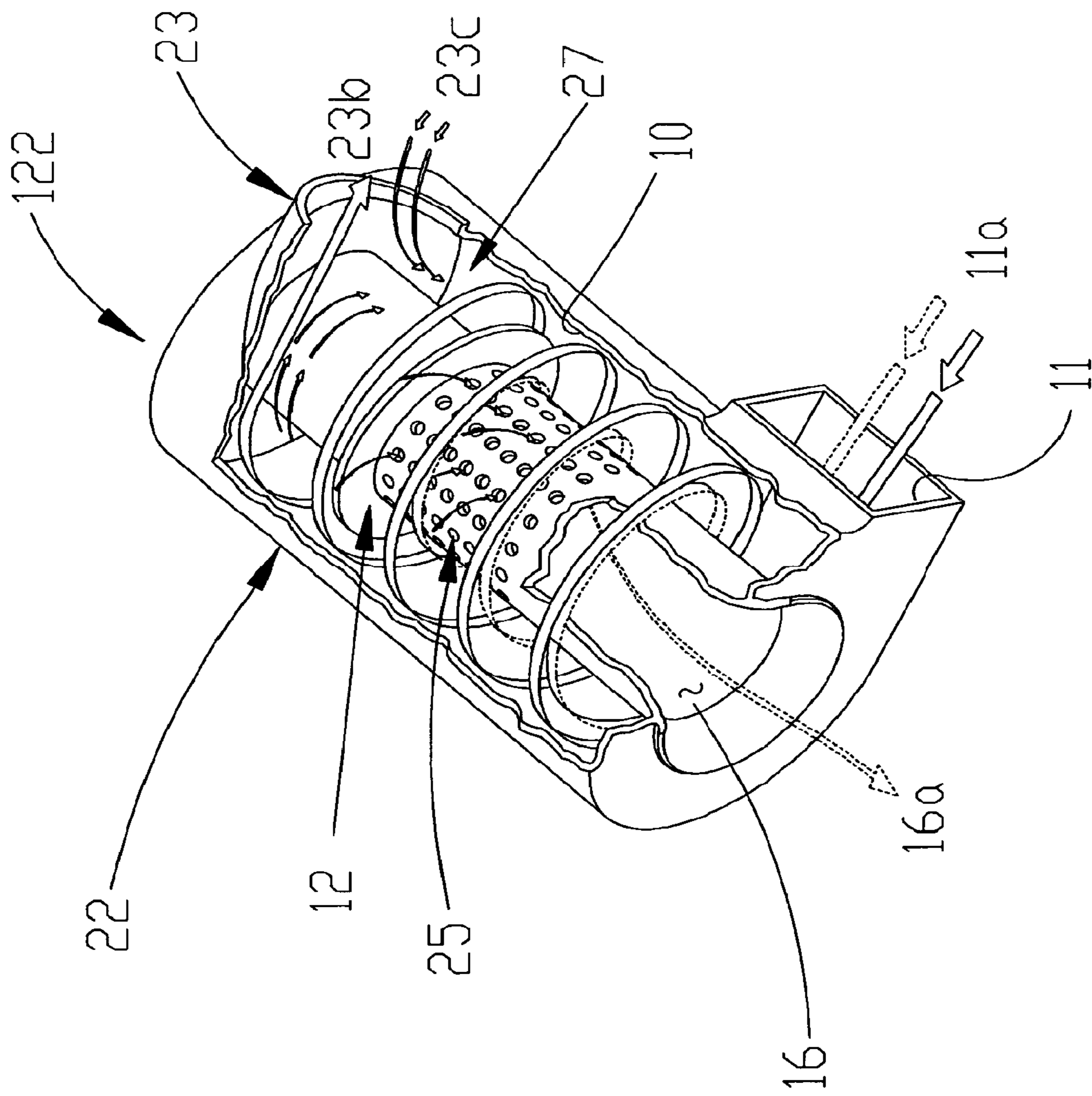


Figure 12

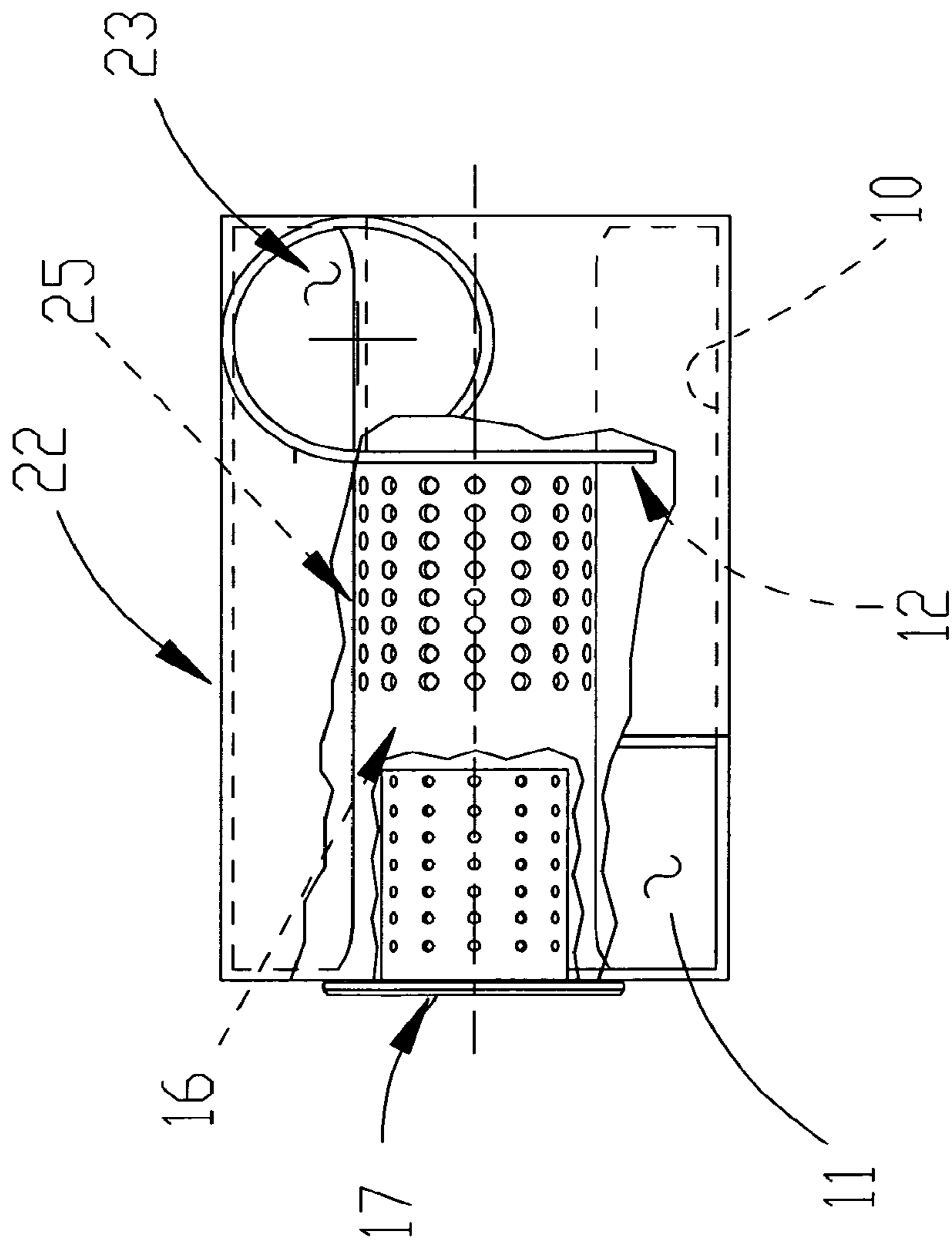


Figure 13



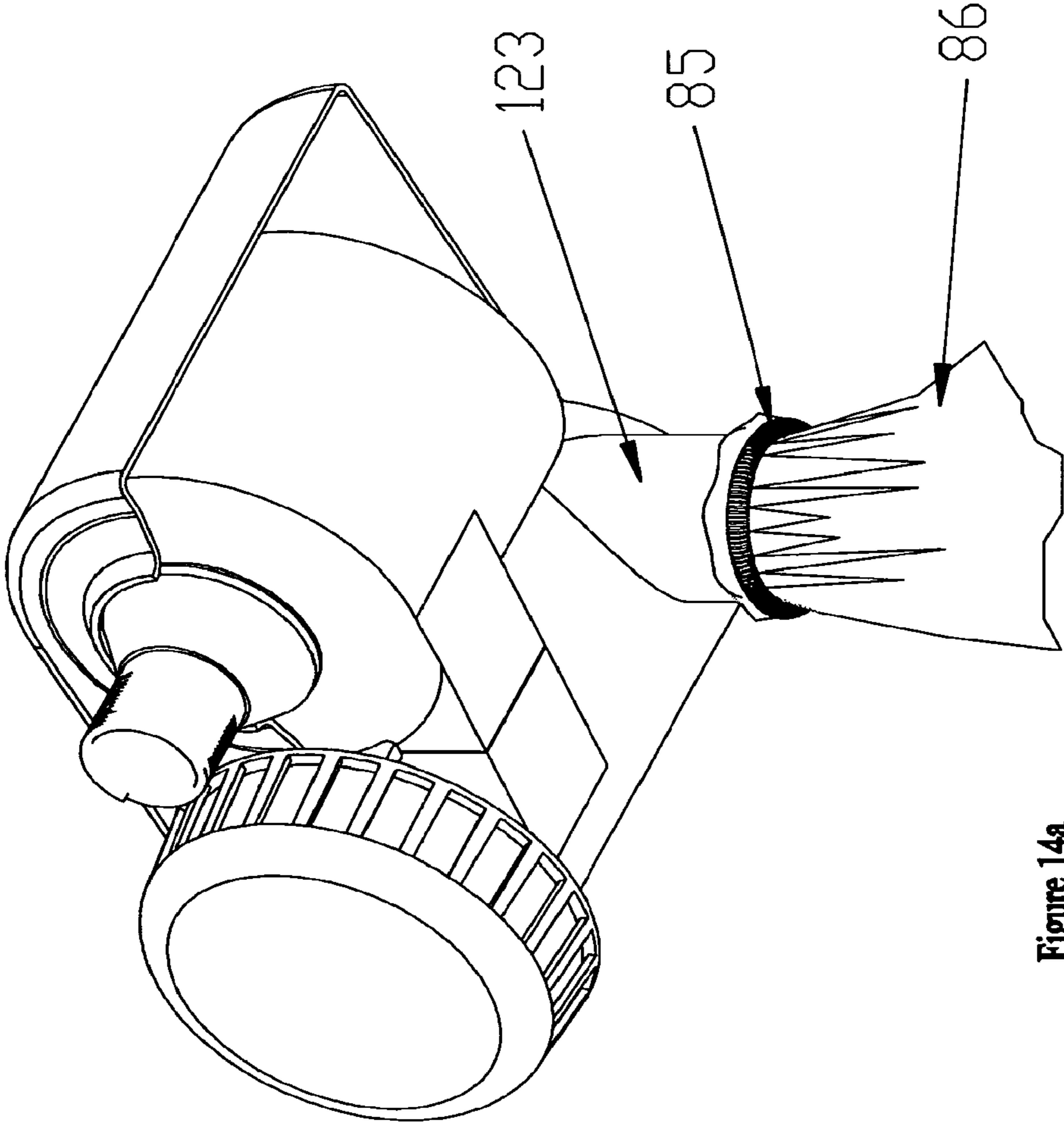


Figure 14a

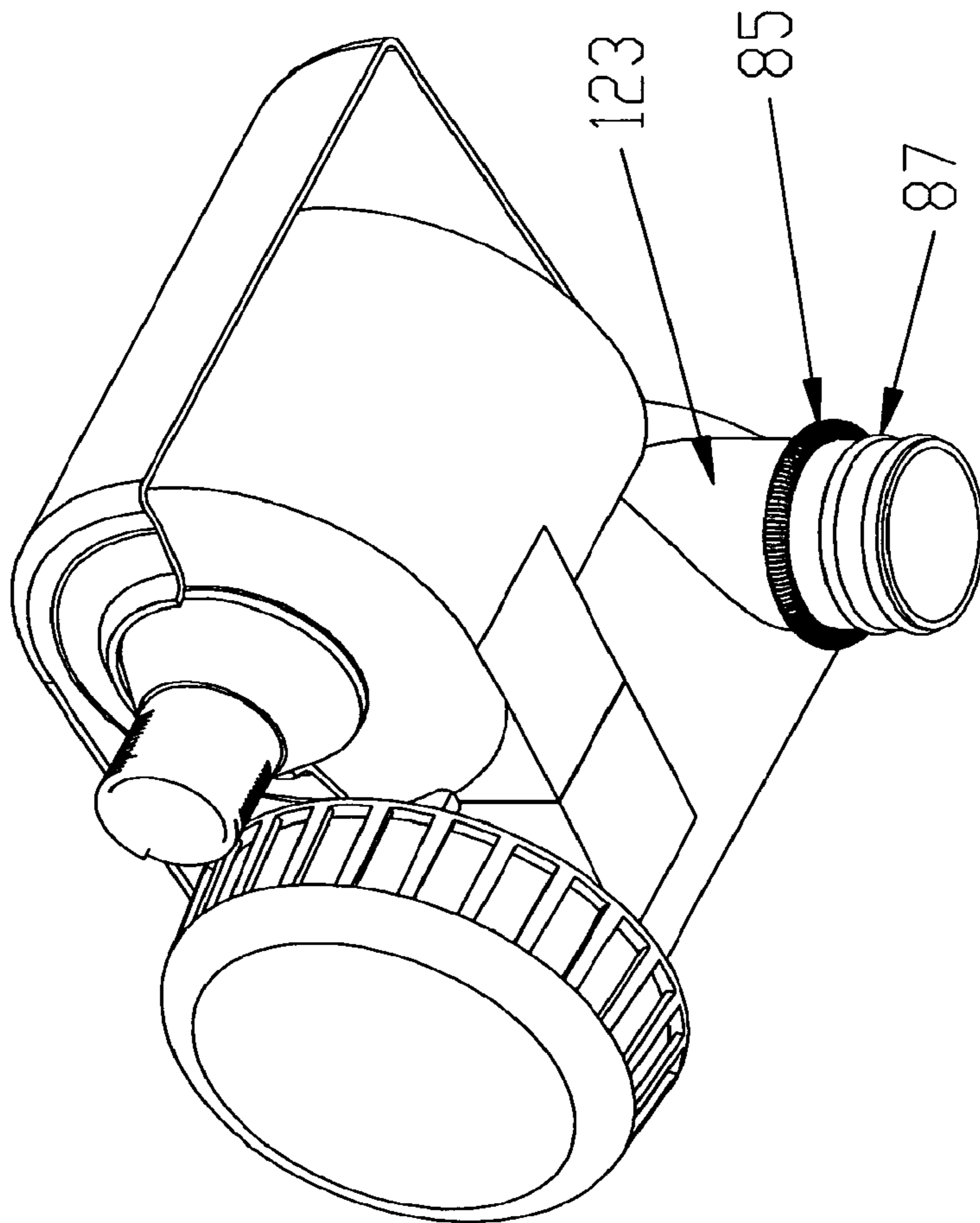


Figure 14b

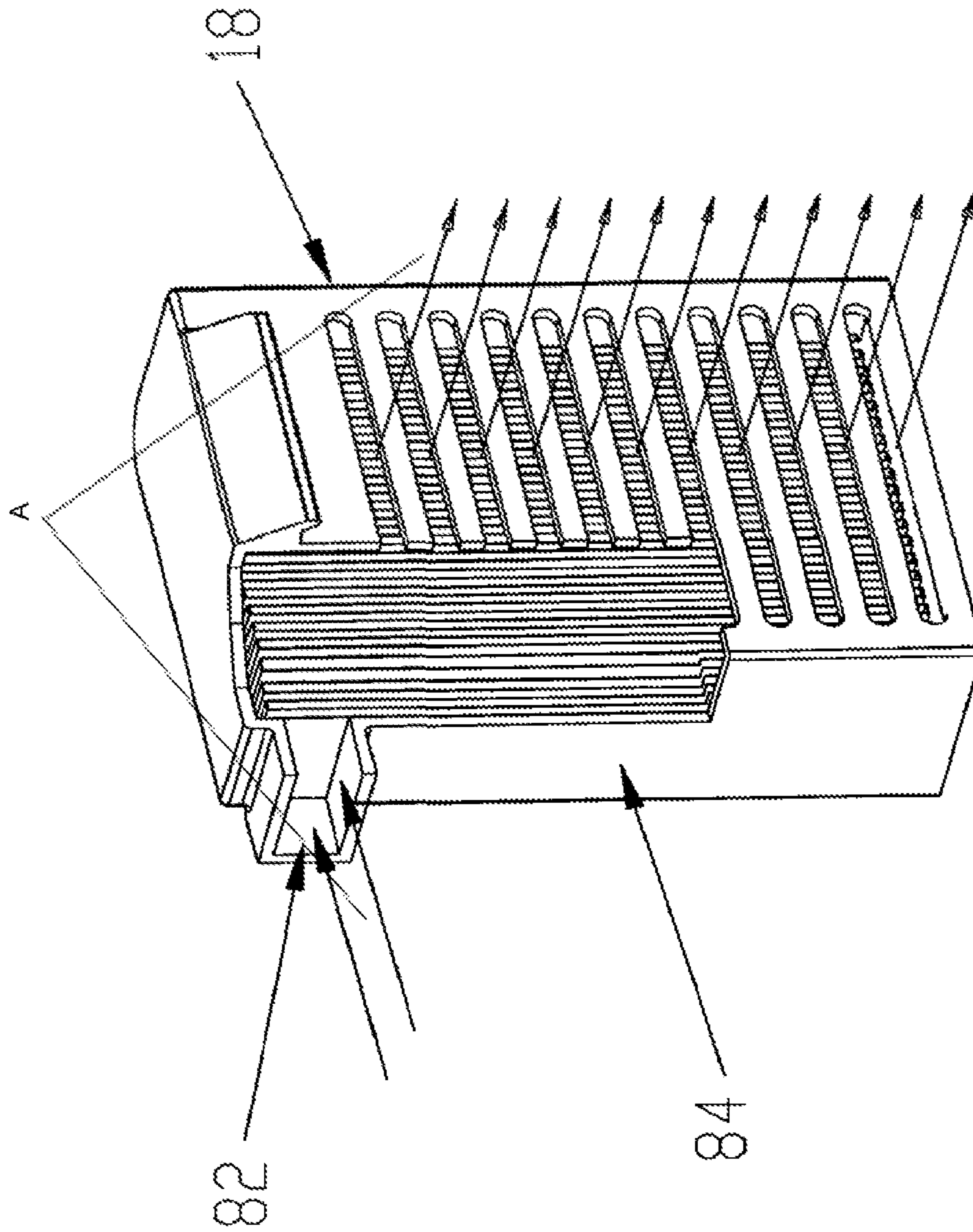


Figure 15

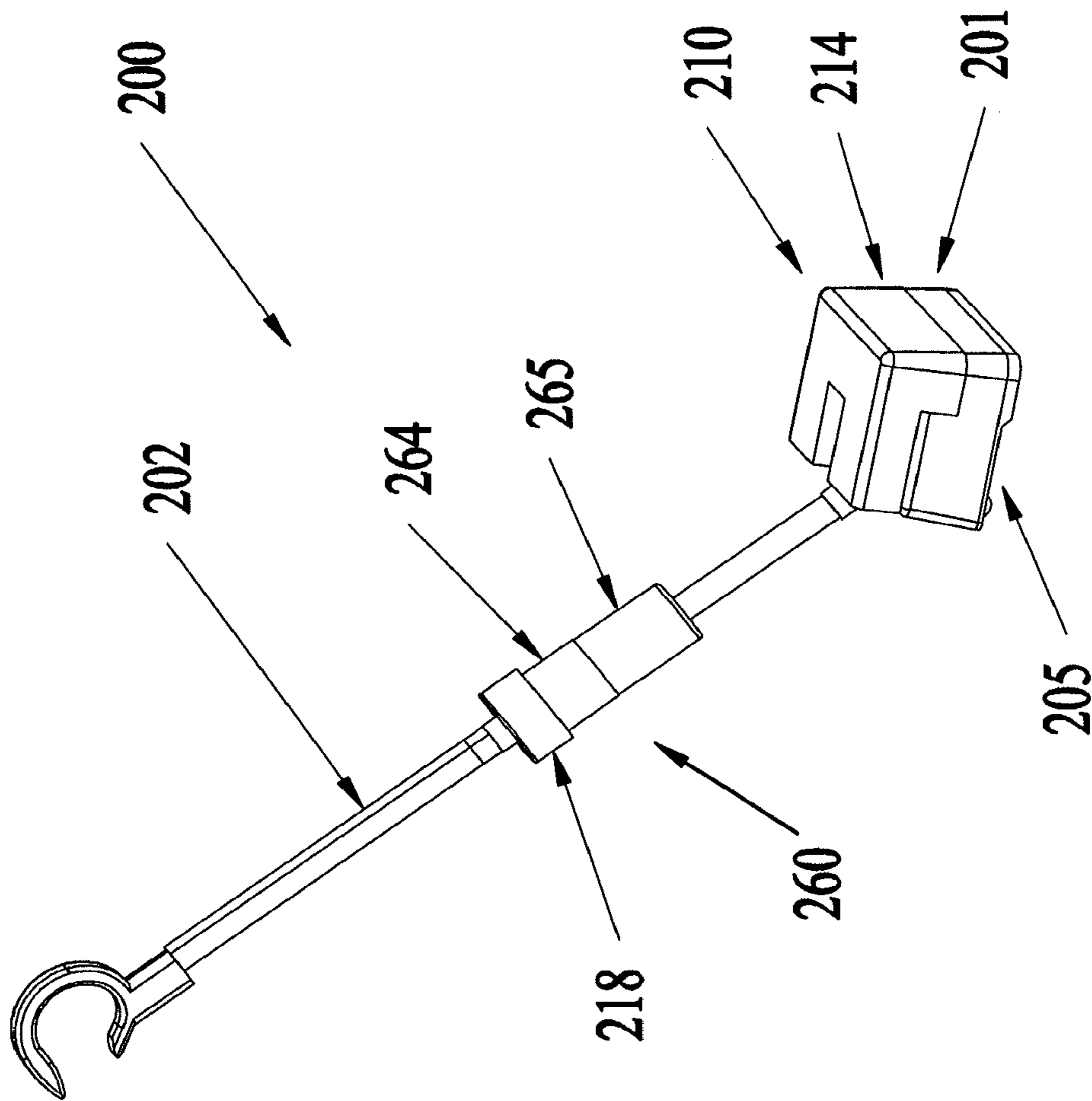


Figure 16

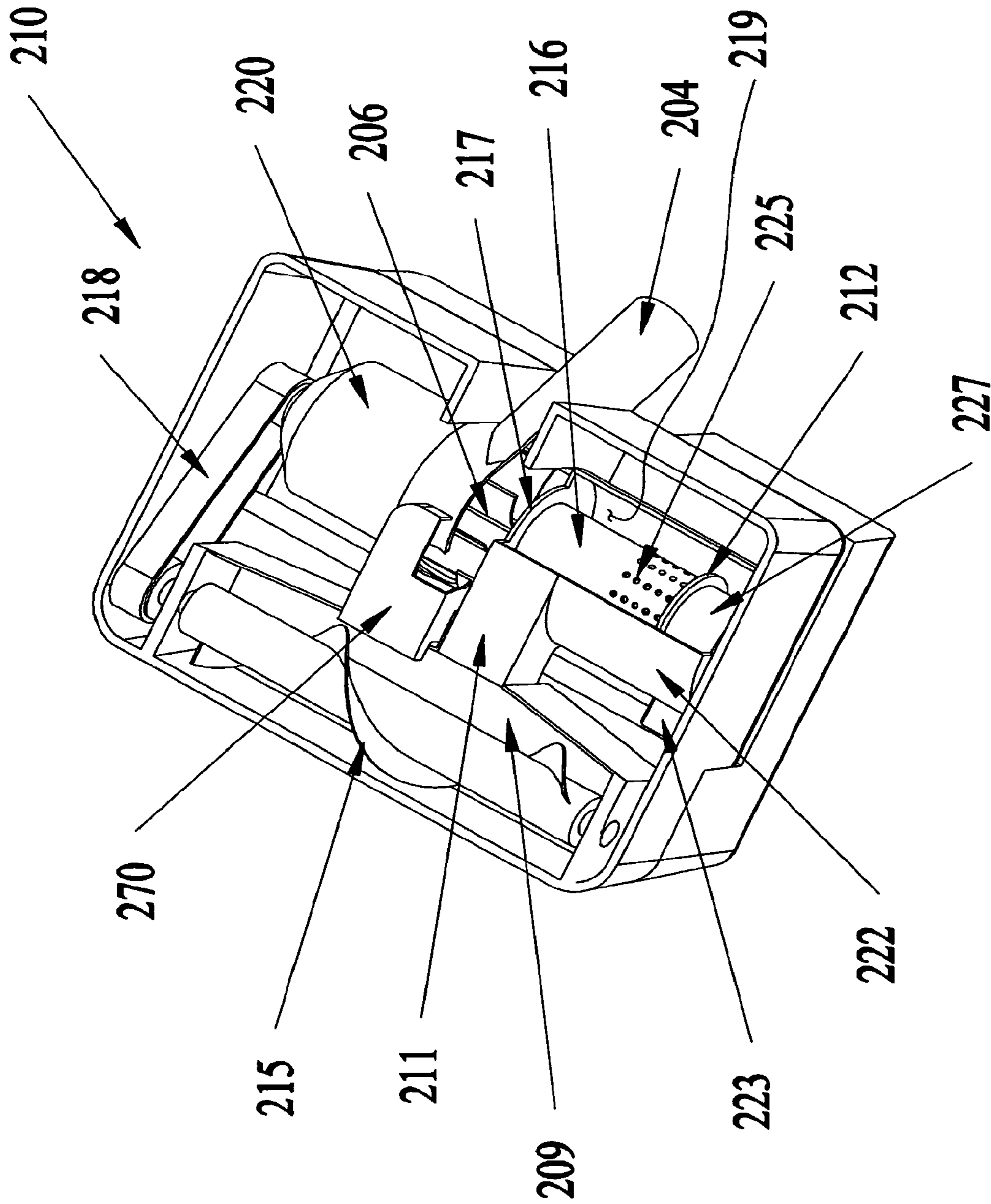


Figure 17



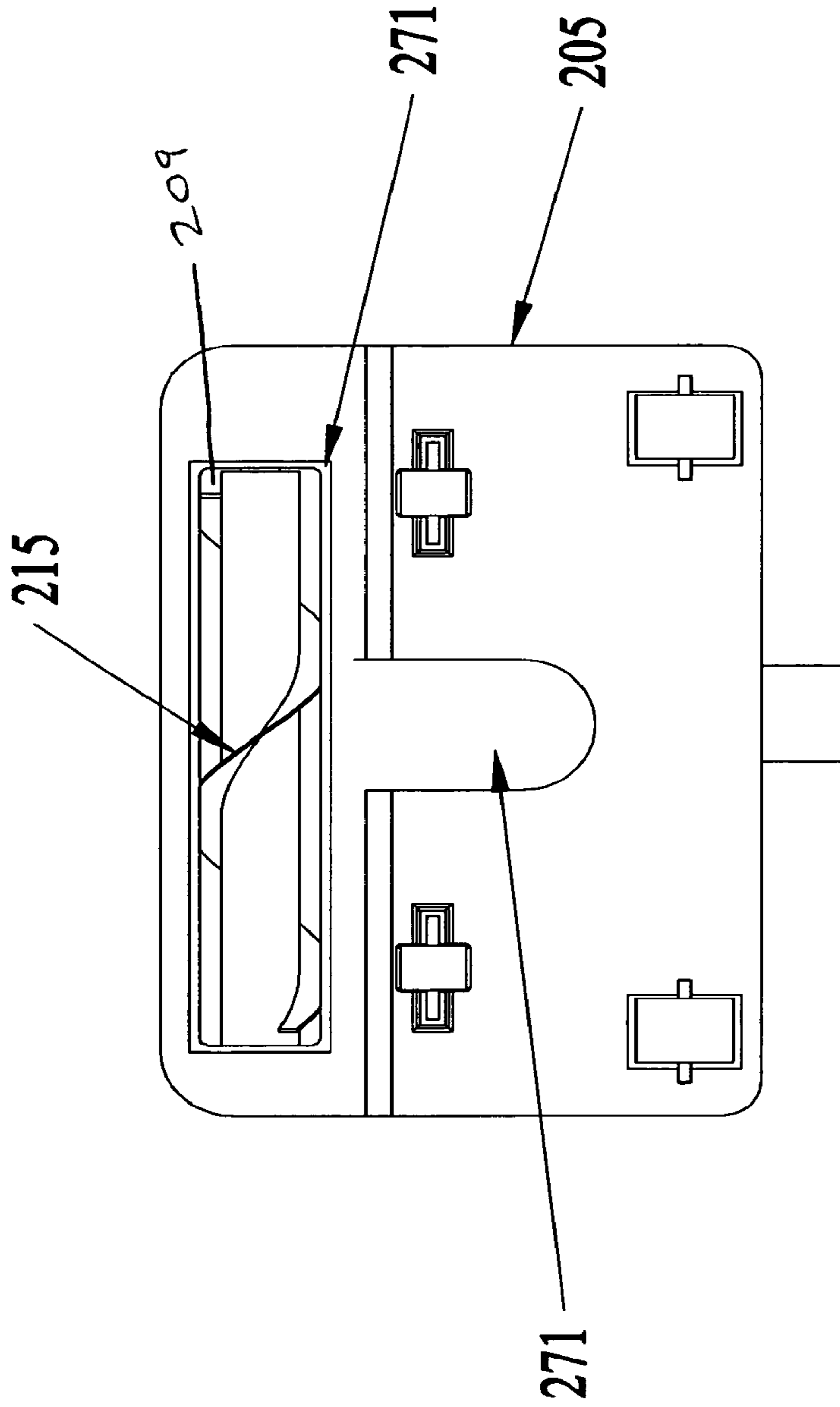


Figure 18

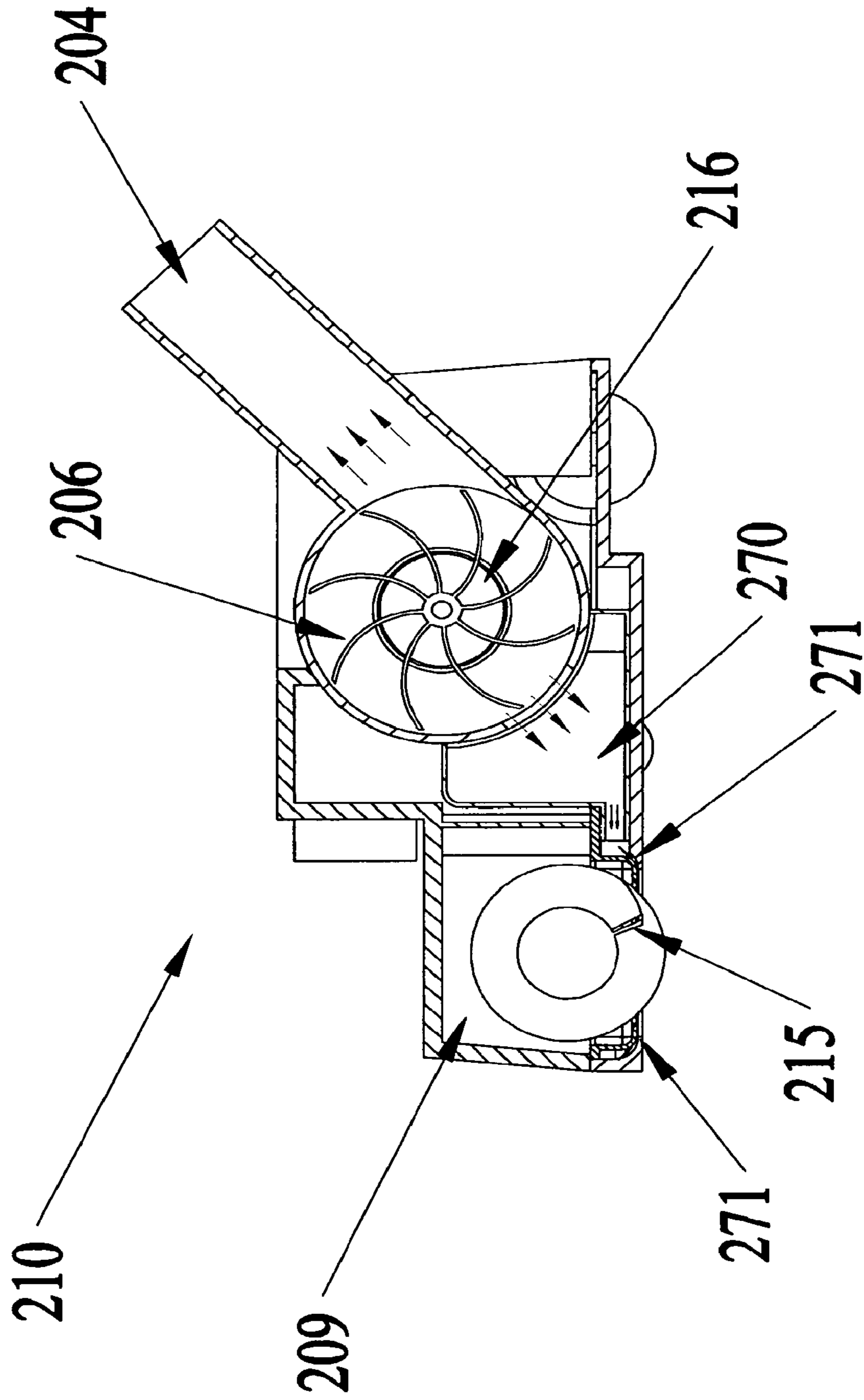


Figure 19

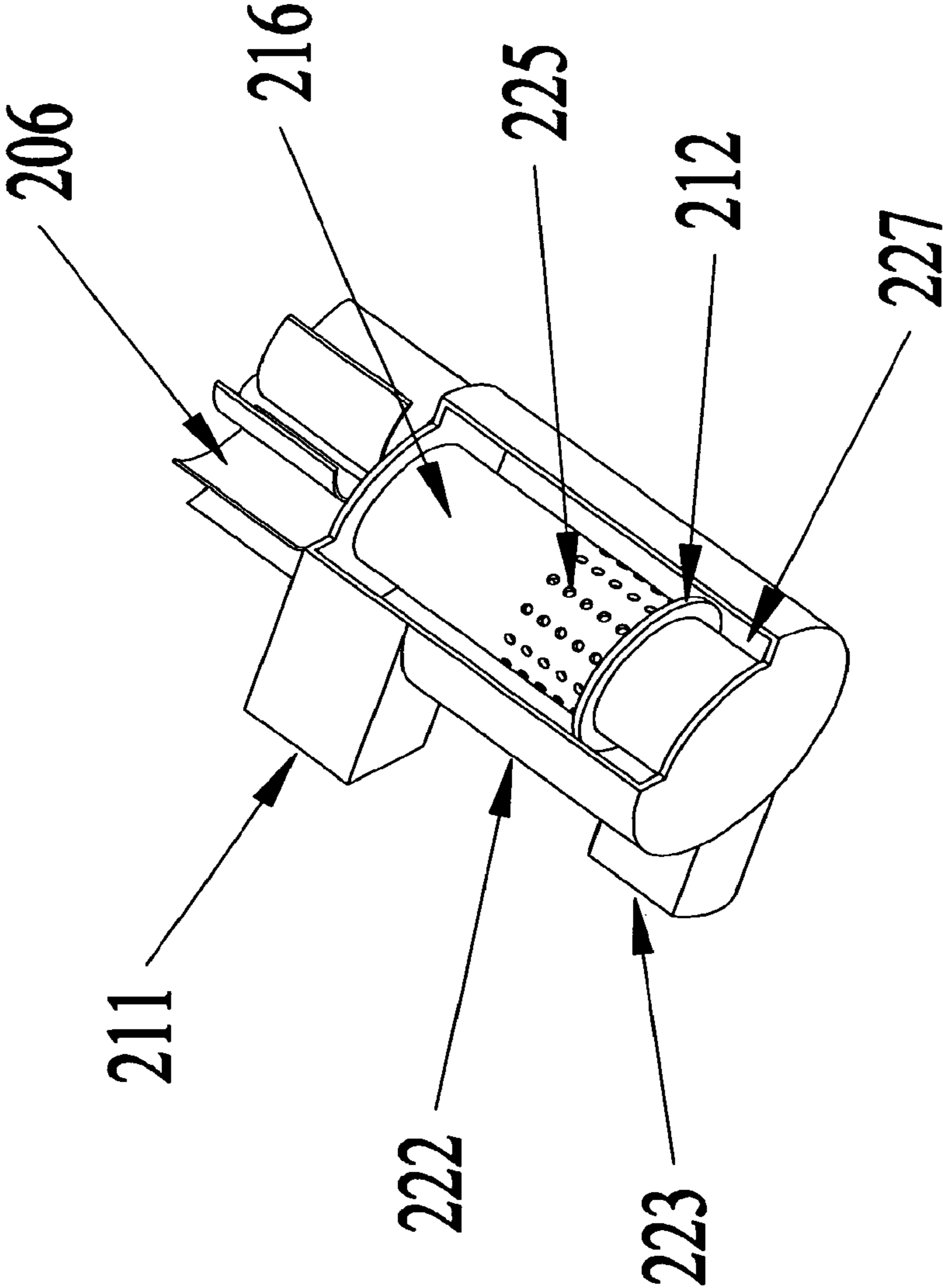


Figure 20

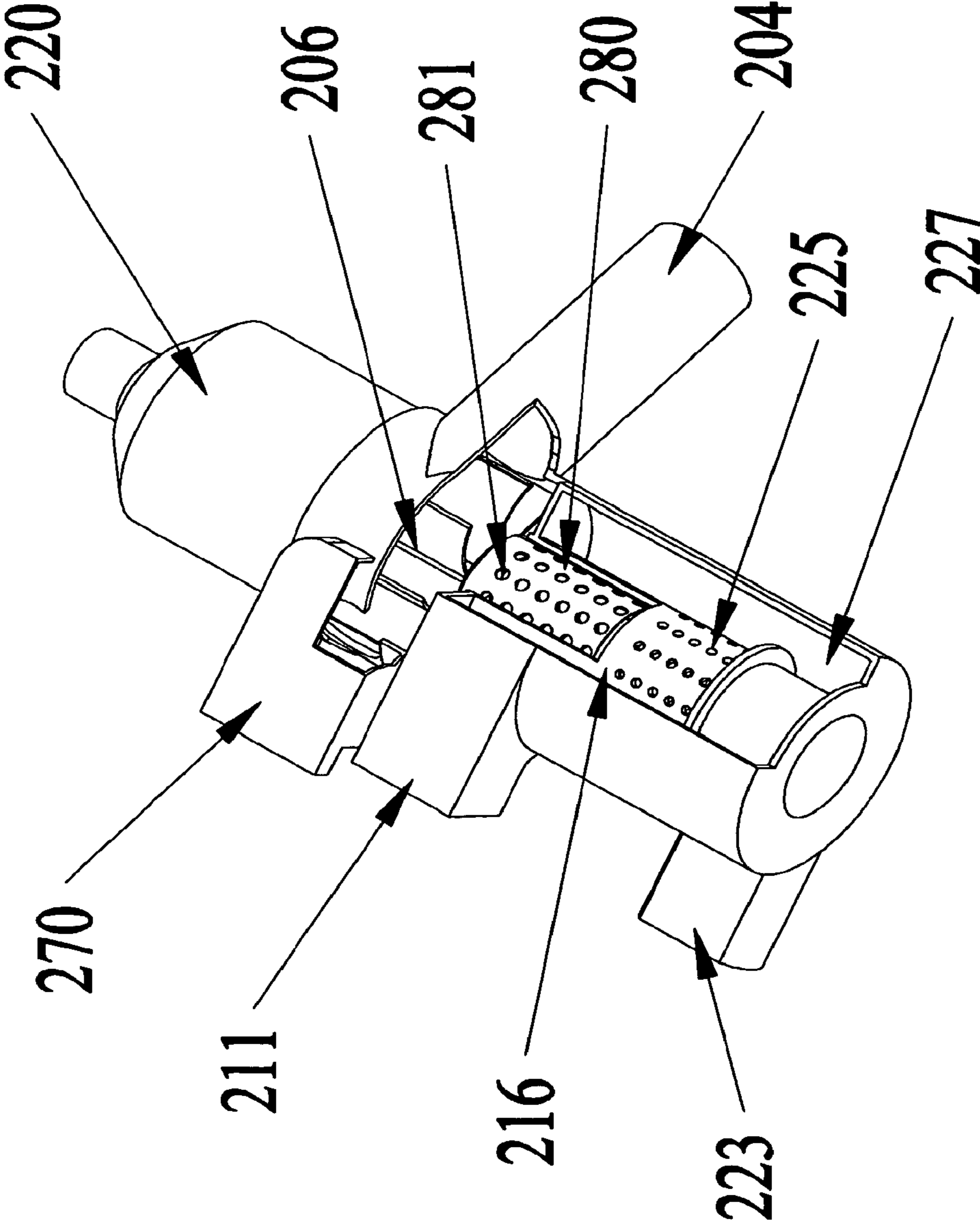


Figure 21

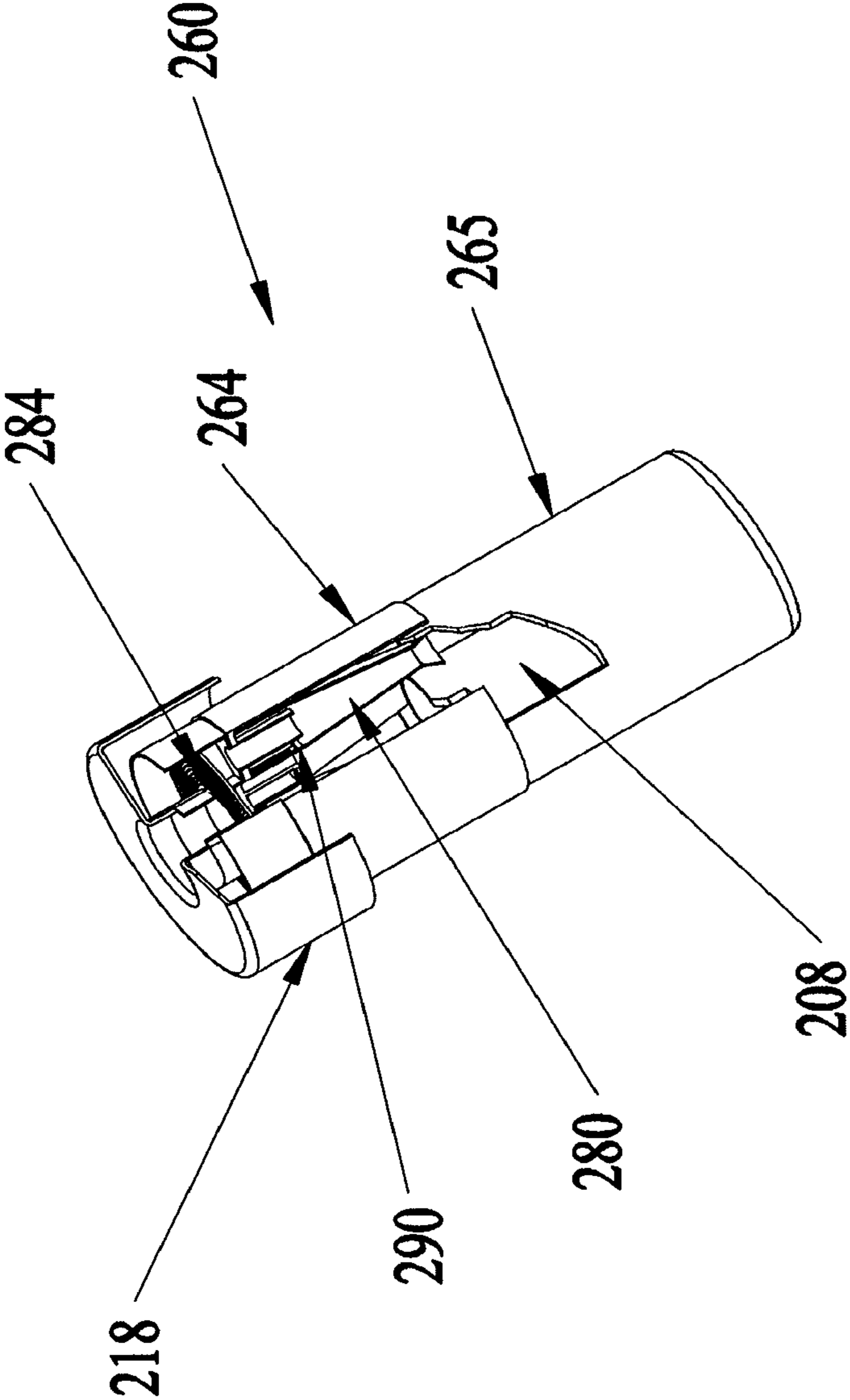


Figure 22



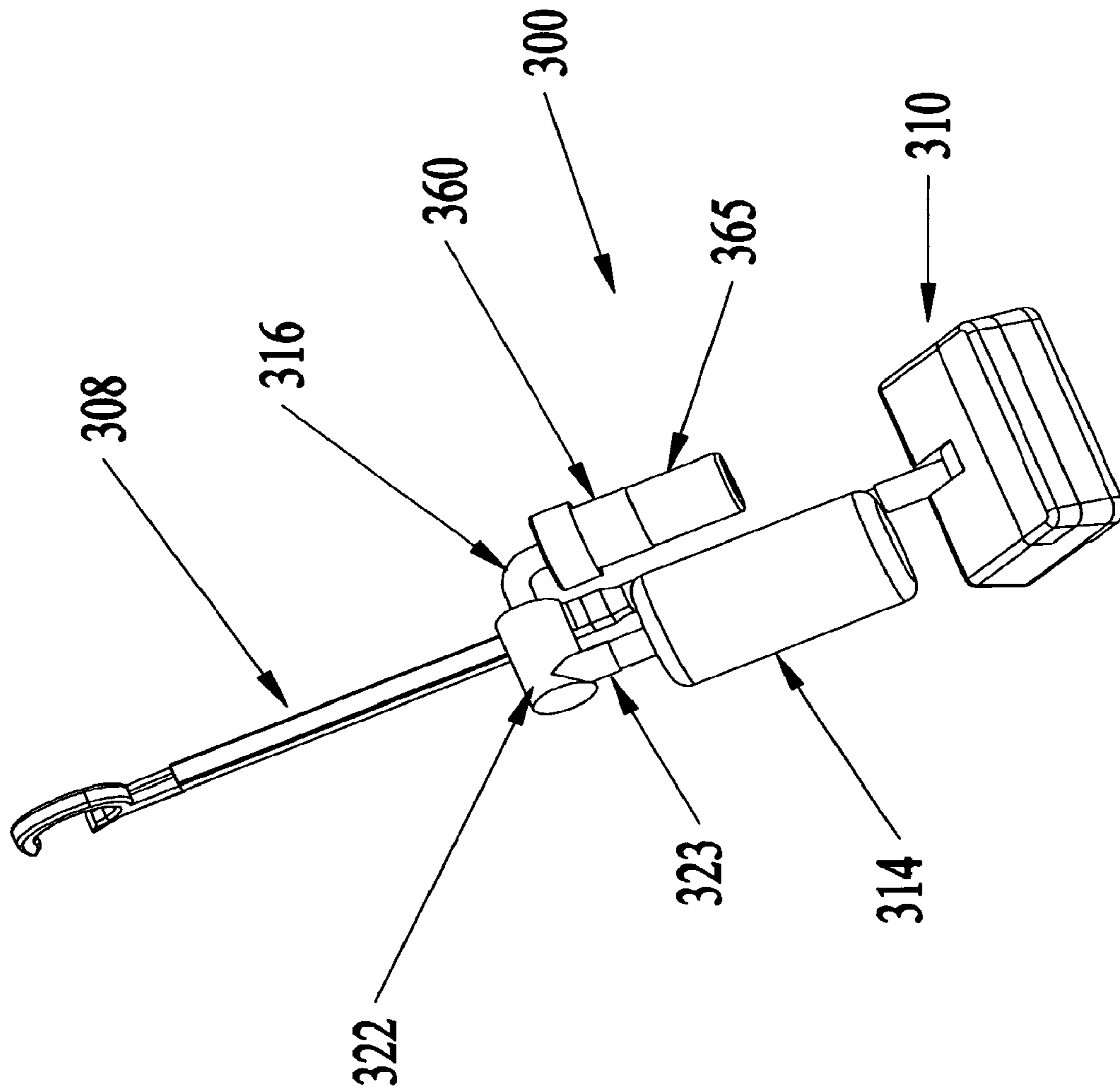


Figure 23

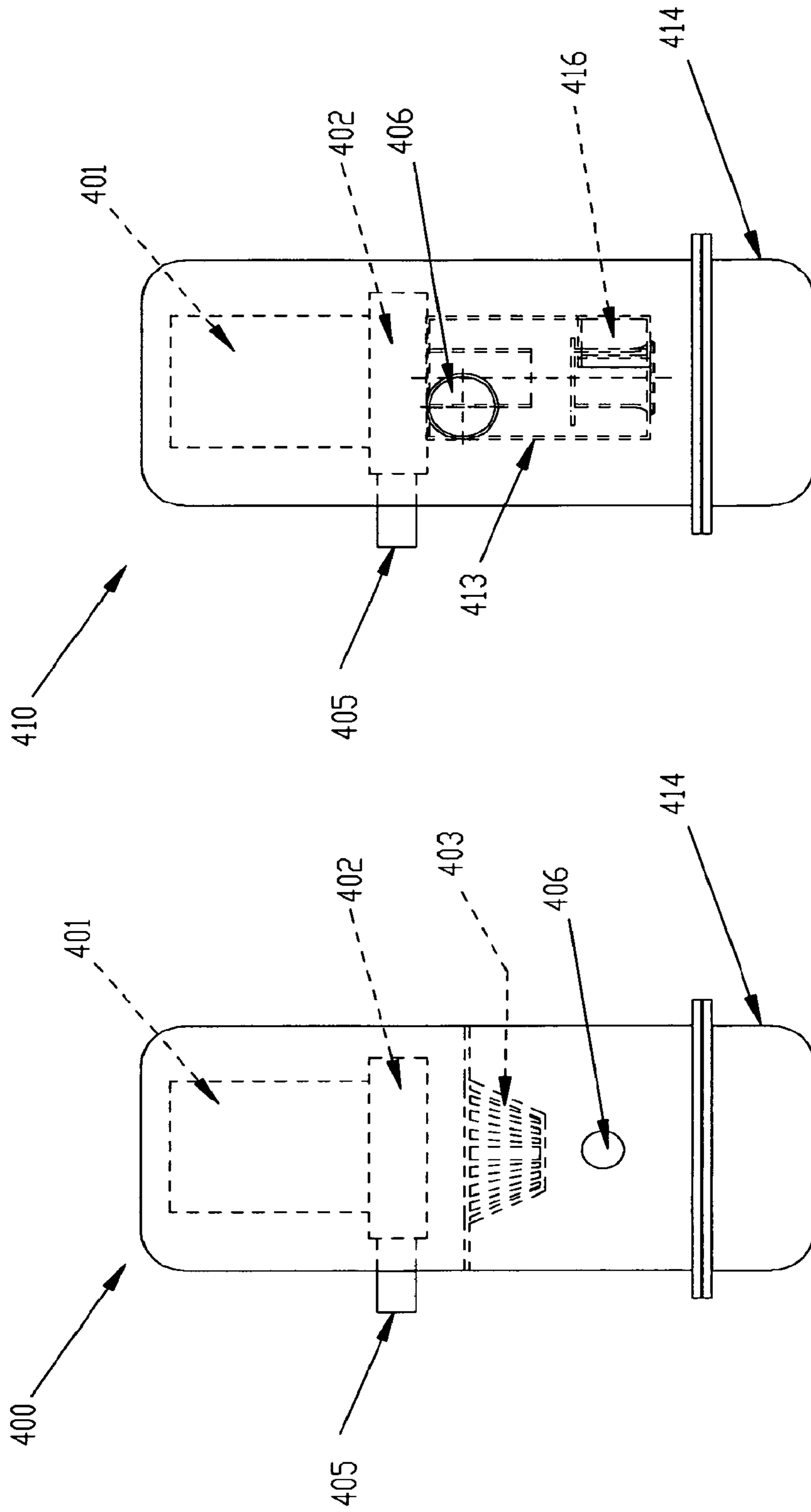


Figure 24

**CENTRIFUGAL DIRT SEPARATION  
CONFIGURATIONS FOR  
HOUSEHOLD-TYPE AND SHOP-TYPE  
VACUUM CLEANERS**

**Matter enclosed in heavy brackets [ ] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue; a claim printed with strikethrough indicates that the claim was canceled, disclaimed, or held invalid by a prior post-patent action or proceeding.**

CROSS-REFERENCE TO RELATED  
APPLICATIONS

The present application is a *Reissue Divisional Application of U.S. Reissue application Ser. No. 15/147,518, filed May 5, 2016, and a Reissue Application of U.S. Pat. No. 9,119,511, issued Sep. 1, 2015, based on U.S. patent application Ser. No. 13/209,867 filed Aug. 15, 2011, which is a continuation of U.S. nonprovisional application Ser. No. 12/074,438 filed Mar. [8] 3, 2008, issued as U.S. Pat. No. 7,996,957 on Aug. 16, 2016* entitled CENTRIFUGAL DIRT SEPARATION CONFIGURATIONS FOR HOUSEHOLD-TYPE AND SHOP-TYPE VACUUM CLEANERS which claims benefit of and priority to U.S. Provisional Patent Application Ser. No. 60/892,723 filed Mar. 2, 2007 entitled CENTRIFUGAL DIRT SEPARATION CONFIGURATIONS FOR HOUSEHOLD-TYPE AND SHOP-TYPE VACUUM CLEANERS, the entire contents of which are hereby incorporated by reference herein.

BACKGROUND

1. Field of the Disclosure

The present application relates to an apparatus for separating dirt or dust particles from an air flow by cyclonic means. The application relates particularly, but not exclusively, to a cyclonic dust separation apparatus for use in a vacuum cleaner.

2. Related Art

Cyclone dust separation devices typically include a frustoconical (truncated cone) cyclone having a tangential air inlet at the one end having a large diameter and a cone opening leading to a dirt or dust collection area at the other end which has a smaller diameter.

There are numerous patents describing a variety of bagless vacuum cleaners now on the market by manufacturers such as Dyson, Hoover, Bissell; i.e. U.S. Pat. Nos. 5,858,038; 5,062,870; 5,090,976; 5,145,499; 6,261,330 and 5,853, 440; English Patent Pub. No. GB727137; and French Patent Pub. No. FR1077243.

U.S. Pat. No. 6,261,330 discloses a device including a fan for causing fluid to flow through the cyclone separator, the cyclone separator having an inlet and an interior wall having a frusto-conical portion tapering away from the inlet, wherein the fan is positioned in the inlet to the cyclone separator chamber on the same axis thereof, such that fluid passing through the fan is accelerated towards the interior wall, and thereby, given sufficient tangential velocity to cause cyclonic separation of particles from the fluid flow within the cyclonic separator chamber. The fan motor is located on the centerline of the cyclone separator chamber, and thus, adds to the size of the cyclone separator chamber.

In U.S. Pat. No. 6,261,330, the inlet port arrangement and the concentric exit port connectors to the cyclone separator

are not optimum. The cyclone chamber depends on gravity to keep the dirt in the bottom of the collection chamber, thus requiring the suggested alternate configuration in which the motor is connected to the fan by a long shaft that extends through the cyclone chamber to the fan at the top of the chamber. This position is not ideal for providing suction to lift dirt from the floor. The patent contends that this is an advantageous design because it lowers the center of gravity of the device as a whole when compared to the embodiment shown with the motor at the top of the vertical cyclone separation chamber.

Since many standard vacuum cleaner motors now run at very high RPM's (22,000 RPM, for example) they provide good airflow and vacuum performance with reduced weight. Having a long shaft through the cyclone separator chamber, however, as suggested by the referenced patent, would not be ideal since shaft critical speed vibration problems are likely to result, thus preventing any weight reduction options to improve the desirability of the vacuum cleaner for the public use.

All of the cyclonic separator type vacuum cleaners now on the market have their cyclone separator chamber on the suction side of the fan so that they are driven by the air flow that is being sucked through them. This has the advantage of only clean air being pulled through the fan impeller, but provides much less velocity and energy than would be available by placing the cyclone separation chamber on the discharge side of the vacuum fan.

Accordingly, it would be desirable to provide a cyclonic dust separation device, preferably suitable for use in a home vacuum cleaner that avoids the problems discussed above.

SUMMARY

It is an object of the present invention to provide an apparatus for separating particles from a fluid flow having a cyclone separator which is efficient, compact, lightweight, and easy to service and maintain.

A cyclonic separation device in accordance with an embodiment of the present application preferably includes a first cyclone chamber having a cylindrical shape with a predetermined diameter, the first cyclone chamber including, a tangential inlet positioned on a first longitudinal end of the first cyclone chamber, a baffle plate positioned in the first cyclone chamber a predetermined distance from the tangential inlet, a tangential dirt outlet positioned on a second end of the cyclone chamber, opposite the inlet and on an opposite side of the baffle plate from the tangential inlet and a center exit duct mounted in the center of the cyclone chamber having an inlet opening positioned upstream from the baffle plate such the centrifuged fluid without particles flows into the center exit duct and out of the cyclone chamber.

The cyclonic separation device of the present application may be used in a variety of applications, including, but limited to use in centrifugal separation type vacuum cleaners.

A vacuum cleaner in accordance with an embodiment of the present invention preferably includes a handle and a floor housing to which the handle is pivotally connected. The floor housing preferably includes a suction fan motor, a suction fan driven by the motor and including a plurality of fan blades driven at a high velocity by the suction fan motor to suck a fluid from a first side of the fan to the second side of the fan, a pick up head positioned adjacent to a floor and in fluid communication with the suction fan and a cyclonic separator device. The cyclonic separator device includes a



first cyclone chamber having a cylindrical shape with a predetermined diameter, the first cyclone chamber including a tangential inlet positioned on a first longitudinal end of the first cyclone chamber, a baffle plate positioned in the first cyclone chamber a predetermined distance from the tangential inlet, a tangential dirt outlet positioned on a second end of the cyclone chamber, opposite the inlet and on an opposite side of the baffle plate from the tangential inlet; and a center exit duct mounted in the center of the first cyclone chamber having an inlet opening positioned upstream from the baffle plate such the centrifuged fluid without particles flows into the center exit duct and out of the first cyclone chamber, wherein the pick up head and suction fan are connected in fluid communication with the first cyclone chamber such that fluid flows from the pick up head through the tangential inlet into the first cyclone chamber and rotates therein at high velocity such that particles in the fluid are forced out to the inner surface of an outer wall of the first cyclone chamber and beyond the baffle plate to be discharged through the dirt discharge outlet.

A vacuum cleaner in accordance with another embodiment of the present invention preferably includes a handle and a floor housing to which the handle is pivotally attached, The floor housing preferably includes a suction fan motor, a suction fan, driven by the motor, a first cyclone separator connected to an inlet of the suction fan. The first cyclone separator preferably includes a first cyclone chamber having a cylindrical shape with a predetermined diameter, the cyclone chamber including a tangential inlet positioned on a first longitudinal end of the first cyclone chamber, a baffle plate positioned in the chamber a predetermined distance from the tangential inlet, a tangential dirt outlet positioned on a second end of the cyclone chamber, opposite the inlet and downstream of the baffle plate, a center exit duct mounted in the center of the cyclone chamber having an inlet opening positioned downstream from the baffle and in fluid communication with the suction fan inlet such that rotation of the suction fan draws fluid into the first cyclone chamber to rotate at high velocity forcing particles in the fluid past the baffle plate and out of the tangential dirt outlet and a removable dirt collector in fluid communication with the tangential dirt outlet and structured to store the particles discharged from the tangential dirt outlet.

Other features and advantages of the present invention will become apparent from the following description of the invention which refers to the accompanying drawings

#### BRIEF DESCRIPTION OF THE DRAWING(S)

FIG. 1 illustrates an upright floor sweeper vacuum cleaner in accordance with an embodiment of the present application

FIG. 2 shows a top view of the vacuum cleaner described for FIG. 1.

FIG. 3 shows a front view of the conceptual configuration of FIG. 1.

FIG. 4 shows a cross sectional side view of the upright floor sweeper of FIG. 1.

FIG. 5 shows a schematic type top view of the vacuum shown in FIG. 4.

FIG. 6 shows a perspective schematic type view of a basic pressure driven cyclone separator for use with the vacuum cleaner of FIG. 1.

FIG. 7 shows a schematic type side view of a basic pressure driven compact cyclone separator for use with the vacuum cleaner of FIG. 1.

FIG. 8a shows a partial cross sectional perspective flow drawing of a bank of small diameter cyclone separators for a secondary separator for use with the vacuum cleaner of FIG. 1.

FIG. 8b is a top view schematic of FIG. 8a.

FIG. 9 shows a cross section of a floor sweeper upright type vacuum including the secondary separator of FIG. 8.

FIG. 10a illustrates an alternative embodiment of a vacuum cleaner in accordance with the present invention.

FIG. 10b illustrates another alternative embodiment of a vacuum cleaner in accordance with the present invention.

FIG. 11a shows the vacuum cleaner of FIG. 10a with a bottom dirt collector removed.

FIG. 11b shows the vacuum cleaner of FIG. 10b with a bottom dirt collector removed.

FIG. 12 shows an alternative embodiment of a primary cyclone separator of the vacuum cleaner of FIG. 1.

FIG. 13 shows a side view of an embodiment of a primary cyclone separator for the vacuum cleaner of FIG. 1.

FIG. 14a shows an alternate configuration of the vacuum cleaner of FIG. 10.

FIG. 14b shows the vacuum cleaner of FIG. 14a with the dirt bag removed and the retention spring rolled back.

FIG. 15 is an exemplary illustration of a HEPA type very small particle filter for use with the vacuum cleaner of the present application.

FIG. 16 illustrates another exemplary embodiment of a vacuum cleaner in accordance with the present application.

FIG. 17 shows a bottom view of the vacuum cleaner of FIG. 16 without a bottom cover.

FIG. 18 shows an external view of the bottom of the vacuum cleaner of FIG. 17 with the bottom cover in place.

FIG. 19 shows cross sectional side view of the vacuum cleaner of FIGS. 16-18.

FIG. 20 shows a more detailed view of the primary cyclone separator of the vacuum cleaner of FIGS. 16-19.

FIG. 21 shows an alternative embodiment of the primary cyclone separator of FIG. 20.

FIG. 22 shows a secondary cyclone separator suitable for use in the vacuum cleaner of FIG. 16.

FIG. 23 illustrates a vacuum cleaner in accordance with another embodiment of the present application.

FIG. 24 shows an illustration of a high performance cyclone separator of the concept illustrated in FIG. 6 applied to replace a cleanable filter cloth in a central vacuum system in accordance with an embodiment of the present invention.

#### DETAILED DESCRIPTION OF EMBODIMENTS

The theory of cyclone dirt or dust separation suggests that efficiency can be increased by increasing the tangential velocity of the air in the separation chamber. This would typically suggest providing a more powerful motor to create a higher rate of fluid flow. However, there are limits to the size and weight of motors that the market will tolerate for domestic vacuum chambers, since the size and weight of these chambers naturally influences the size and weight of the resulting domestic home vacuum cleaner as a whole. Increased complexity and size also add to the cost of the vacuum cleaner, which is also an important consideration in the competitive home vacuum cleaner market.

Thus, reducing the size of the motor required to provide a simple high efficiency domestic home vacuum cleaner or shop vacuum cleaner is very desirable. Smaller, lighter weight, more energy efficient vacuum cleaners provide significant advantages in such a competitive market. The vacuum cleaner of the present application allows for a



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reduction in motor size in that it preferably provides the dirt separation chamber on the output, or blowing, side of the suction fan, which allows the suction fan to impart more speed to the dirt laden air as it is provided to the separation chamber. Thus, higher speed air is provided in the separation chamber without the need to use a larger motor. In addition, the diameter of the separation chamber may also be reduced, which also aids in maintaining high velocity air flow therein and provides better separation while reducing overall vacuum size. These features are described in further detail below.

In the vacuum cleaner **1** (See FIG. **1**), for example, of the present application, the design of the fan motor and fan can be separately optimized while maintaining proper motor cooling. Further, the vacuum cleaner of the present application is preferably buildable using existing highly developed domestic vacuum cleaner motors now in production. In addition, the incoming flow to the fan impeller does not have to be compromised and the cyclonic separating chamber can be optimized separately without the need to compromise its design for motor or fan considerations, as is the case in the prior art discussed above.

The design of the present application minimizes the opportunities for flow passage blockage that is a problem in other cyclone dirt separation vacuum cleaners now on the market since all of the air flow elements are preferably close connected with minimum duct work and high velocity air. This reduces the opportunity for velocity and pressure drops as air flows through the cleaner.

FIG. **1** illustrates a conceptual perspective view of an upright type vacuum floor cleaner **1** in accordance with an embodiment of the present application. The cleaner **1** includes a handle assembly **4** which can be pivotally mounted to the side of the vacuum cleaner housing assembly **5** which is partially carried by rear wheel assembly **7** and whose pick up head **9** rides in close proximity to a carpet or floor. The head area **9** preferably includes small rollers (not shown) mounted under the housing **5** as well.

The vacuum cleaner suction fan and motor assembly **20** generates suction that is connected to the head area **9** of the vacuum cleaner **1**. The suction lifts dirt and dust from the floor and into the vacuum **1**. This dirt-laden air then passes through the motor driven fan **6** (See FIG. **4**) and is accelerated by the high velocity of the fan rotor blades **8**. In a preferred embodiment, the velocity of the blades **8** may be almost the speed of sound (1100 ft/sec) such that the air and dirt is thrown through the tangential input connecting duct **11** to the primary cyclone separator **22**. The primary cyclone separator **22** preferably includes a relatively small diameter cyclone chamber **10** (preferably approximately 4 inches in diameter) where the dirt is moved against the outside walls by the very high centrifugal forces and passes the baffled plate **12** (See FIGS. **6** and **7**) to be discharged tangentially from the chamber **10** through tangential dirt outlet **23** into a dirt collection bag or container **14**. *The arrows A included in FIGS. 4, 8a, 9 and 15 illustrate the flow of air through the vacuum cleaner 1.*

The dirt free air, however, moves towards the center of the cyclone chamber **10** and exits through a central duct **16** where it can then be finally filtered by filter **18**, if desired, or run through a secondary cyclone separator **65** which preferably includes a group of small diameter cyclone chambers **60** which generate very high g-forces due to their smaller diameter.

It is preferred that the air velocity remain high and that the components of the cleaner **1** are closely coupled together to

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provide for minimum pressure drop between components and to maintain a very open flow design.

The secondary cyclone separator **65** is shown in more detail in FIGS. **8a** and **8b**, and is preferably embodied as a group of small diameter tangential entry chambers **60** on top of truncated cones **62** that taper to a decreased radius for increasing centrifugal force and including truncated opening **63** at the bottom thereof to provide for dirt discharge into a separate, very fine dirt collection chamber **64**. This chamber **64** can also be separately cleaned less often than the larger dirt collection chamber **14**. The dirt exit, or openings **63** of each of the small truncated cones **62** can have a reverse cone shape to spread the spinning dirt outwardly and allow more separation between the discarded dirt and the returning air circulation at this location. Air preferably enters the chambers **60** via the inlets **61**.

The primary, first, cyclone chamber **10** removes all of the larger dirt and a large part of the smaller dirt because of its high velocity, before the air is discharged into these small diameter chambers **60** through connecting duct openings which allow them to operate at maximum efficiency. Thus, the primary cyclone chamber **10** effectively deals with the larger, more voluminous dirt by discharging it into a large collection container **14** which can be several times the capacity of the low efficiency cyclone first stage chamber of bagless vacuum cleaners now on the market since they have to capture the large dirt in the lower part of their cyclone chamber and provide sufficient space to accommodate dirt storage and cyclonic separation. In contrast, in the cleaner **1**, for example, of the present application, the dirt is discharged tangentially from the primary cyclone chamber into a separate container for dirt storage. Thus, the size of the primary cyclone chamber is reduced and this provides improved efficiency. Dirt storage can be increased as well, since a separate chamber is provided for the separated dirt, this chamber can be rather large which allows the chamber to be emptied less often. It is noted that the dirt collection chamber **14** is preferably removably attached to the cleaner **1** to allow it to be easily removed and emptied.

The secondary cyclone section **65** has high efficiency and includes a plurality of small diameter cyclone chambers **60** which are left to function in their optimum condition with comparatively clean air, i.e. air only including particles with a diameter of 50 microns.

The air can then be withdrawn centrally from each of the second stage high efficiency cyclones chambers **60** via the ducts **70** and exhausted, if desired, through exit duct **80** to HEPA filter **18**, if desired. However some, or most of this air may alternatively be returned to the vacuum pick up head **9** through the opening **13** to provide jet assisted suction at the pick up area (FIG. **9**). The opening **13** is preferably positioned to discharge the returned air substantially parallel to the floor, creating an area of low pressure just above the floor due to the high velocity of the returned air. This area of low pressure (i.e. Bernoulli pressure) aids in suction at the pick up head **9**. The return air is in turn sucked back into the vacuum **1** again where the cycle is repeated. In this manner, the air sucked into the cleaner **1** can be recycled to aid in further suction and separation.

The primary cyclone separation chamber **10** provided in the cleaner **1** of the present application preferably has a relatively small diameter (4 inches, for example) which is quite small when compared to that required when the dirt is being separated on the suction side of the vacuum fan, as in the prior art discussed above. This allows for a much more compact, lighter weight and lower manufacturing cost vacuum cleaner. Also, the configuration of the cleaner **1**



ensures that the dirt is not captured at the bottom of the primary cyclone chamber, but is discharged tangentially into a bag or dirt compartment **14** separated from the cyclone chamber **10**. This, as previously stated, also allows for a reduction of the size of the cyclone separation chamber and more versatility to allow the cyclone dirt separation chamber to be used in a variety of vacuum cleaners configurations including shop vacuums or canister type vacuums, as well as carpet sweeper uprights such as that illustrated in FIG. **1**.

The vacuum cleaner design of the present application also has many advantages over prior art vacuum cleaners that use disposable porous bags which must be purchased separately and require frequent replacement. These bag-type vacuums lose effectiveness as the filter bags becoming full and fine particles become trapped by the filter bag to degrade its permeability and cause a loss of suction. While vacuum cleaners using cyclonic separation chambers are known in the art and avoid the problems of replaceable bag cleaners discussed above, these cyclone separation vacuum cleaners are very large, since they must accommodate the larger separation chambers necessary to provide separation and dirt storage.

One of the important features of the vacuum cleaner described herein is to provide for open air flow and to separate the dirt from the air by intense centrifugal force cyclone action such that filtration is only a final back-up if necessary at all.

In a preferred embodiment, the vacuum cleaner **1** of the present application preferably includes a pick up head **9** with a power driven carpet brush **3** (See FIG. **4**). The suction fan and motor assembly **20** preferably includes an electric motor and impeller, impeller inlet and fan **6** with a tangential discharge outlet that is aligned with the tangential inlet **11** of the primary cyclone separation chamber **10** which has cylindrical walls. The dirt collection chamber **14** may be embodied as a simple non-porous bag or a separate chamber and is connected to a tangential outlet **23** of the chamber **10**. A back up filter **18** may be provided as well, if desired. A secondary cyclone separator **65** may also be provided in the discharge flow path of the primary separator chamber **10**. This secondary separator **65** is preferably optimized to remove fine particles from the air.

Referring to FIG. **2** which is a top view looking down on the vacuum housing assembly **5** of the cleaner **1** of FIG. **1**, the air flow path from the dirt pick up head **9** through the suction fan and motor assembly **20** and into the primary cyclone dirt separator chamber **10** can be seen.

FIG. **3** shows a front view of the cleaner **1** of FIG. **1**. FIG. **4** shows a cross sectional side view of the upright floor sweeper cleaner **1** of FIG. **1** showing the direct tangential close coupled connection between the suction motor and fan rotor assembly **20** and the tangential inlet **11** to the cyclone separator chamber **10** along with the brush **3** (add to FIG. **4**). The brush **3** may be driven by a belt connected to the fan **6** or motor shaft (not shown). FIG. **5** shows a schematic type top view of the vacuum shown in FIG. **4**. FIG. **6** shows a perspective schematic type view of the primary cyclone chamber **10** for use with the vacuum cleaner of FIG. **1**. In particular, FIG. **6** illustrates the exit duct **16** which allows cleaned air to exit the chamber **10**. It is noted that the separation chamber **10** of FIG. **6**, for example may be used in a variety of applications including various vacuum cleaner configurations with the same benefits. FIG. **7** shows a schematic type front view of the primary cyclone separator chamber **10** with tangential inlet **11** and dirt outlet **23** and with the central air exit passage **16** with inlet **24**. The baffle

plate **12** separates the tangential dirt discharge area proximate the outlet **23** from the recirculation area of the chamber **10**.

FIG. **12** illustrates an alternative embodiment of the primary cyclone separation section **22** in which the center central air exit duct **16** include an inlet **24**, as illustrated in FIG. **7** covered by a perforated cylinder including a plurality of small diameter holes **25** (i.e. 0.076-0.2 inches) rather than being fully open. The holes **25** provide noise isolation and prevent any large dirt or fluff from carpet being discharged from the chamber **10** during any periods of pressure fluctuation, i.e. momentary pressure fluctuations when the vacuum moves from a carpet to a bar floor.

In operation, the dirt-laden air enters tangential inlet **11** as shown by the airflow lines **11a**. The dirt is moved to the outer walls of the chamber **10** by the centrifugal force resulting from the high velocity of the inlet dirty air and the relatively small diameter of the chamber **10**. The centrifugal dirt separation force may be determined based on the following equation:

$$F=w/gv^2/r$$

where "F" represent the centrifugal force, "w" represents the weight flow, g is a gravitational constant, "v" is the velocity of the air and "r" is the inside radius of the chamber **10**. The dirt particles move down the chamber **10** and pass the baffle plate **12** to be discharged from the chamber **10** at high velocity out of tangential outlet **23**. The outlet **23** is preferably connected to the collection chamber **14**, or to a bag to collect the dirt. The lighter air that accompanies the dirt into the chamber **14** is recirculated back as is illustrated by the line **23c** of FIG. **12** and into the chamber, or swirl area **27** downstream of the baffle **12** and recirculated in this area. The air flow exits the cyclone separator chamber through the holes **25** in the duct **16**. This exit air is very clean due to the high centrifugal force in the chamber **10**, which separates particles from the airflow. Only the clean air near the center of the chamber **10** is allowed to exit.

FIG. **13** shows a side view of the primary cyclone separator chamber **10** with an additional perforated liner duct, or insert, **17** inserted into the duct **16** to provide sound (noise) dampening. The duct **17** is designed to provide a Helmholtz resonator effect due to its hole sizes and cavity spacing behind the liner walls to reduce the noise emitted from the cleaner **1**, for example.

FIG. **10a** shows a conceptual perspective view of a canister type or shop vacuum cleaner **100** in accordance with another embodiment of the present application. The cleaner **100** preferably includes a top cover and frame **102** on which the basic vacuum cleaner elements may be mounted, including, suction fan motor and fan assembly **120**, centrifugal separator **122**, and final filter **108**. A vacuum hose (not shown) may be attached to the suction fan inlet **110**. Further, there is preferably a carrying handle **125** provided along with a lower dirt collecting housing **114**.

FIG. **11a** illustrates cleaner **100** of FIG. **10a** without the dirt collecting housing **114** such that the mounting of the basic components **120**, **122**, **118/108** can be seen as well as the tangential inlet **111** to the centrifugal separator **122** and the suction fan tangential discharge port **116** as well as the tangential dirt discharge port **23** of the centrifugal separator **122**.

In FIG. **10b**, the suction fan inlet **110** is shown connected to the inside top area of **120** of the primary dirt collection chamber **114a** where the inlet **110b** to the vacuum cleaner **100b** is moved to enter the primary dirt collection chamber **114a**. This positioning allows nails or other large items of



debris commonly cleaned using a shop vacuum to be collected before the air passes through the fan. In addition, if the vacuum 100b is used to pick up water, for example, the majority of the water will be trapped in the main container 114a before complete separation is achieved by the primary cyclone separation chamber. Thus, the collector chamber preferably includes low pressure side 114a and a fan discharge pressure side 114b that collects dirt or water separated from the suction air and discharged from the separator 122 out tangential discharge outlet 123 into the chamber 114b.

The design of FIG. 10b represents a much improved shop vacuum (or wet pick-up shop vacuum) which typically only clean air with a washable sponge or cloth filter such that the discharged air is often very dusty. Similarly, when liquid is picked up, the discharge air tends to be very wet since the filter is saturated by water still in the air that is passing through the discharge opening.

FIG. 11b shows the under side of the vacuum top assembly of FIG. 10b with the container 114 removed. The motor and suction fan inlet 110 is now relocated inside the vacuum cover 120 to provide suction by inlet 110 directly into the container portion 114a.

FIG. 14a illustrates an alternate configuration of the vacuum cleaner 100 of FIG. 10a with a non-porous plastic or paper bag 86 attached to the cyclone chamber's tangential discharge 123. The throw away bag 86 is preferably held in place with a roll spring 85 which can be rolled over the bag opening to clamp it to the tangential dirt discharge duct. FIG. 14b shows the vacuum cleaner 100 with the dirt bag 86 removed and the retention spring 85 rolled back to expose slot 87 which is preferably formed on the outlet 123 to accommodate the spring 85 to keep the bag 86 in place. The shop vacuum cleaner of FIGS. 10b and 11b may also utilize a bag as well to collect discharge dirt, if desired. The bag may be positioned in, or in place of the chamber 114b, if desired.

FIG. 15 is a partial sectional view of the HEPA type very small particle filter 18 that is shown on the upright floor sweeper vacuum cleaner 1 of FIG. 1, or on the alternative embodiment of FIG. 22, discussed below. The filter 18 preferably receives air discharged from the secondary cyclone separator 65 in FIG. 1, or to the air discharge duct of the primary or secondary separator sections of the embodiment of FIG. 22. The filter 18 provides for final air filtration if desired. The filter 18 preferably includes a housing 84 and an inlet 82 into which the cleaner air from the primary and secondary cyclone separation sections 22, 65 pass for final filtering.

FIG. 16 illustrates a compact, light weight cyclone (centrifugal) dirt separator, bagless re-circulated air and sound suppressed vacuum cleaner 200 in accordance with an embodiment of the present application. The vacuum cleaner 200 preferably includes a suction fan drive motor 220, fan 206, a large dirt centrifugal separator section 222 connected to the suction fan inlet 217 and a large collection chamber 214, where all of these components are mounted in the floor housing 201. See also FIG. 17.

A handle 205 is preferably pivotally attached to the housing 201. A secondary cyclone separator section 260 is preferably mounted on the handle 205, which is at least partially hollow to allow air to flow from housing 201 to the separator 264. A second removable dirt collector 265 is provided with the secondary separator 264 which is for very fine dirt and need only be cleaned periodically. In addition, a HEPA filter 284 may also be provided to provide additional final filtering, if desired, as shown in FIG. 22.

FIG. 17 shows an internal perspective view of the housing 201 with a bottom cover removed such that the major components are visible. As illustrated, the primary cyclone separator section 222 is mounted adjacent to the suction fan motor and housing 220. The fan 206 rotates to create suction and pull dirt and air from the pick up head area 209 through the tangential inlet 211 and into the cyclone chamber 219 of the separator 222. The dirt rotates in the chamber 210 at high velocity and moves to the inner surface of the outer walls of the chamber and past the baffle 212 into the discharge area 227 from which it is discharged through tangential outlet 223 into the removable large dirt collector or bag 214 shown in FIG. 16. A belt 218 is preferably connected to a shaft of the motor or fan and is used to rotate brush 215 in the pick up head area 209 to help lift dirt off the floor. An exit duct 216 is positioned in the chamber 219 to allow the cleaned air to exit the chamber through the holes 225 formed in a wall therein. The duct 216 is connected to the fan inlet at 217. Element 227 refers to the dirt swirl section, or collection section, of the chamber 219 which is downstream of the baffle 212 and includes the tangential dirt discharge outlet 223 for the dirt to be blown into the removable large dirt container 214. Another advantage of discharging the dirt from the cyclone chamber is that the large dirt collection chamber or bag can take any desired shape to maximize dirt volume storage efficiency.

The suction fan 206 air is discharged into the hollow handle mounting 204 with some or most of it being provided to the collection duct 270 for connection to a jet assist slot 271 (See FIG. 18) in the bottom cover 205. Jet assisted suction is discussed above with reference to the vacuum cleaner 1 of FIGS. 1-9, for example. Generally, the high velocity air produces a low-pressure area just above the carpet or floor due to the Bernoulli effect.

FIG. 18 shows a bottom view of the vacuum cleaner 200 of FIG. 17 with the bottom cover replaced. In addition, a recirculation air jet assist slot 271 around the suction pick-up opening 209 is shown with the rotating floor brush 215.

FIG. 19 illustrates a cross sectional view of the housing 201 illustrating how a portion of the cleaned air from the chamber 10 can be redirected to the jet assist slot 271 of the head area 209 while other air is directed up the hollow handle portion 204 to the secondary separator 265.

FIG. 20 is a schematic view of the centrifugal dirt separator section 222 and suction fan 206. As illustrated, the exit air duct 216 of the chamber 210 is connected to the inlet of the fan 206. FIG. 20 also illustrates the relationship of the tangential inlet 211 of the chamber 210 and the tangential dirt outlet 223 as well as the openings 225 that are preferably formed to provide an inlet for the exit duct 216 to allow the cleaned air to escape chamber 222. The suction fan 206 is shown attached to the centrifugal separator exit duct 216 so as to provide noise isolation from the intake of the vacuum cleaner 200 near the suction head area 209.

FIG. 21 is an improvement on the features illustrated in FIG. 20. In this embodiment, a second insert 280 provided in the air exit duct 216 to provide Helmholtz dampening of sound. This absorbs the high velocity fan blade and high velocity air noise from coming back out the inlet 211.

FIG. 22 illustrates the secondary cyclone separator 260 mounted on the handle 205 of the cleaner 200. The separator 264 is optimized for separating very small particles from the cleaned air provided from the primary separator 222. The separator chamber 264 thus includes a plurality of small diameter chambers 290 similar to the chambers 60 described above with reference to FIGS. 8a and 8b. The chambers include small tangential inlets and tapered walls but are



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arrange around the handle 204. Slots are provided in the handle 205 to correspond to these inlet slots. The cup 265 is provided for dirt collection and is preferably removable. In one embodiment a disposable bag may be placed into the cup 265 to collect dirt.

FIG. 23 illustrates an alternative embodiment of a vacuum cleaner 300 where the primary cyclone separator 322 is mounted on hollow handle 308 and the larger dirt and much of the very small dirt is deposited into a non-porous bag or container 314. The container 314 may be made larger in this embodiment since it is not part of the floor assembly. Secondary cyclone separation is provided in the separator 360, which may also include a HEPA filter, if desired. The first and second separators 322, 360 however are similar to those described above with reference to vacuum 200.

FIG. 24 shows the application of the disclosed cyclone separator illustrated in FIGS. 6 and 12, for example, in place of the cleanable filter 403 commonly used in central vacuum systems. Generally, in conventional systems such as system 400 dirt is sucked into a removable container 414 as shown in FIG. 24, so it can be discarded. However, the air is typically filtered by a cloth bag or other cleanable filter (see element 403, for example) which is dusty to clean and reduces performance of the system as it gets clogged with dirt and dust.

In accordance with the present application, the central vacuum 410 has element 401 which represents a suction fan drive motor, and element 402 representing the suction fan while the cyclone separator is identified as element 413 which can be used to replace the filter 403 in the housing of a central vacuum cleaner 400. The inlet port 406 from the central home vacuum is connected to the house vacuum piping which is connected to the tangential inlet of the separator 413. A center air discharge duct similar to duct 16 of FIG. 6 is preferably connected to the suction fan inlet 402 to allow the suction fan to draw air at high velocity through the tangential inlet of the cyclone centrifugal separator 413. The separated dirt is discharged out tangential discharge 416 and drops into the container 414.

Although the present invention has been described in relation to particular embodiments thereof, many other variations and modifications and other uses will become apparent to those skilled in the art. It is preferred, therefore, that the present invention be limited not by the specific disclosure herein, but only by the appended claims.

What is claimed is:

[1. A vacuum cleaner, comprising:

a handle; and

a floor housing to which the handle is pivotally connected, wherein

the floor housing further comprises:

a suction fan motor;

a suction fan driven by the suction fan motor and including a plurality of fan blades driven at a high velocity by the suction fan motor to suck a fluid from a first side of the fan to a second side of the fan;

a pick up head positioned adjacent to a floor and in fluid communication with the suction fan; and

a cyclonic separator device comprising:

a cyclone chamber having a cylindrical shape with a predetermined diameter, the cyclone chamber further comprising:

a tangential inlet duct positioned on a first longitudinal end of the cyclone chamber;

an opening formed in an outer wall of the cyclone chamber at a second end of the cyclone cham-

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ber, opposite the inlet through which particles exit the cyclone chamber; and

a center exit duct mounted substantially in the center of the cyclone chamber having an inlet opening positioned such the centrifuged fluid without particles flows into the center exit duct and out of the cyclone chamber, wherein

the pick up head and suction fan are connected in fluid communication with the cyclone chamber such that fluid flows from the pick up head through the tangential inlet into the cyclone chamber and rotates therein at high velocity such that particles in the fluid are forced out to the inner surface of the outer wall of the cyclone chamber and are discharged through the opening.]

[2. The vacuum cleaner of claim 1, wherein the inlet of the center exit duct includes a sleeve including a plurality of perforations formed therein, such that the perforations prevent particles from entering the inlet.]

[3. The vacuum cleaner of claim 1, further comprising a jet assist duct connected between the center exit duct and the pick up head and a jet assist nozzle positioned on the pick up head and connected to the jet assist duct to provide a stream of high velocity air in a direction parallel to the floor to be cleaned to aid in sucking particles off the floor and into the pick up head.]

[4. The vacuum cleaner of claim 3 further comprising a collection chamber in fluid communication with the opening and structured to store separated particles from the fluid.]

5. A canister-type vacuum comprising:

a top cover; and

a bottom particle container portion;

the bottom particle container portion including:

a suction fan motor;

a suction fan driven by the motor and including a plurality of fan blades driven at a high velocity to suck fluid from a first side thereof to a second side thereof;

a cyclonic separation device comprising:

a cyclone chamber substantially cylindrical in shape and having a predetermined diameter; the cyclone chamber further comprising:

a tangential inlet formed on a first longitudinal end thereof and in fluid communication with the second side of the suction fan;

a particle opening formed on a second longitudinal end thereof, opposite the first longitudinal end, and configured for particles to pass out of the cyclone chamber;

a particle container aligned with the particle opening to store particles; and

an exit opening formed in a top of the cyclone chamber, between the first and second longitudinal ends and configured for fluid to leave the cyclone chamber.

6. A canister-type vacuum comprising:

a top cover; and

a bottom particle container portion;

the bottom particle container portion including:

a suction fan motor;

a suction fan driven by the motor and including a plurality of fan blades driven at a high velocity to suck fluid from a first side thereof to a second side thereof;

a cyclonic separation device comprising:

a cyclone chamber substantially cylindrical in shape and having a predetermined diameter; the cyclonic chamber further comprising:

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*a tangential inlet formed on a first longitudinal end thereof;*  
*a particle opening formed on a second longitudinal end thereof, opposite the first longitudinal end and configured for particles to pass out of the cyclone chamber; and*  
*an exit opening formed in a top of the cyclone chamber, between the first and second longitudinal ends and configured for fluid to leave the cyclone chamber, wherein the suction fan is positioned downstream of the exit opening.*

7. A vacuum system comprising:

*a housing;*

*a hose connected to the housing; and*

*a pick up head connected to the hose;*

*the housing further comprising;*

*a suction fan motor;*

*a suction fan driven by the motor and including a plurality of fan blades driven at a high velocity to suck fluid from a first side thereof to a second side thereof;*

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*a cyclonic separation device comprising:*

*a cyclone chamber substantially cylindrical in shape and having a predetermined diameter; the cyclonic chamber further comprising:*

*a tangential inlet formed on a first longitudinal end thereof and in fluid communication with the hose;*

*a particle opening formed on a second longitudinal end thereof, opposite the first longitudinal end and configured for particles to pass out of the cyclone chamber; and*

*an exit opening formed in a top of the cyclone chamber, between the first and second longitudinal ends and configured for fluid to leave the cyclone chamber, wherein the suction fan is positioned downstream of the exit opening.*

8. The vacuum system of claim 7, wherein the hose is made of a flexible material.

9. The vacuum system of claim 7, wherein the hose is a pipe.

\* \* \* \* \*