



US00RE46723E

(19) **United States**
(12) **Reissued Patent**
Smith et al.

(10) **Patent Number: US RE46,723 E**
(45) **Date of Reissued Patent: *Feb. 20, 2018**

(54) **ALIGNMENT RESTORATION DEVICE FOR LOAD TRANSPORTING APPARATUS**

(56) **References Cited**

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1,001,299 A 8/1911 Page
1,242,635 A 10/1917 Anderson

(Continued)

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FOREIGN PATENT DOCUMENTS

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CH 359422 1/1962
CN 1515477 7/2003

(Continued)

(*) Notice: This patent is subject to a terminal disclaimer.

OTHER PUBLICATIONS

(21) Appl. No.: **15/141,587**

Notice of Pending Litigation Under 37 CFR 1.56 Pursuant to Granted Request for Prioritized Examination Under 37 CFR 1.102 €(1), Jun. 28, 2016.

(22) Filed: **Apr. 28, 2016**

(Continued)

Related U.S. Patent Documents

Reissue of:

(64) Patent No.: **9,004,203**
Issued: **Apr. 14, 2015**
Appl. No.: **14/028,150**
Filed: **Sep. 16, 2013**

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U.S. Applications:

(63) Continuation of application No. 13/711,269, filed on Dec. 11, 2012, now Pat. No. 8,561,733.

(Continued)

(51) **Int. Cl.**

B62D 51/06 (2006.01)
B62D 57/02 (2006.01)

(Continued)

(52) **U.S. Cl.**

CPC **B62D 57/032** (2013.01); **B62D 57/02** (2013.01); **E21B 15/003** (2013.01)

(58) **Field of Classification Search**

CPC B62D 57/02; B62D 57/032; E21B 15/003

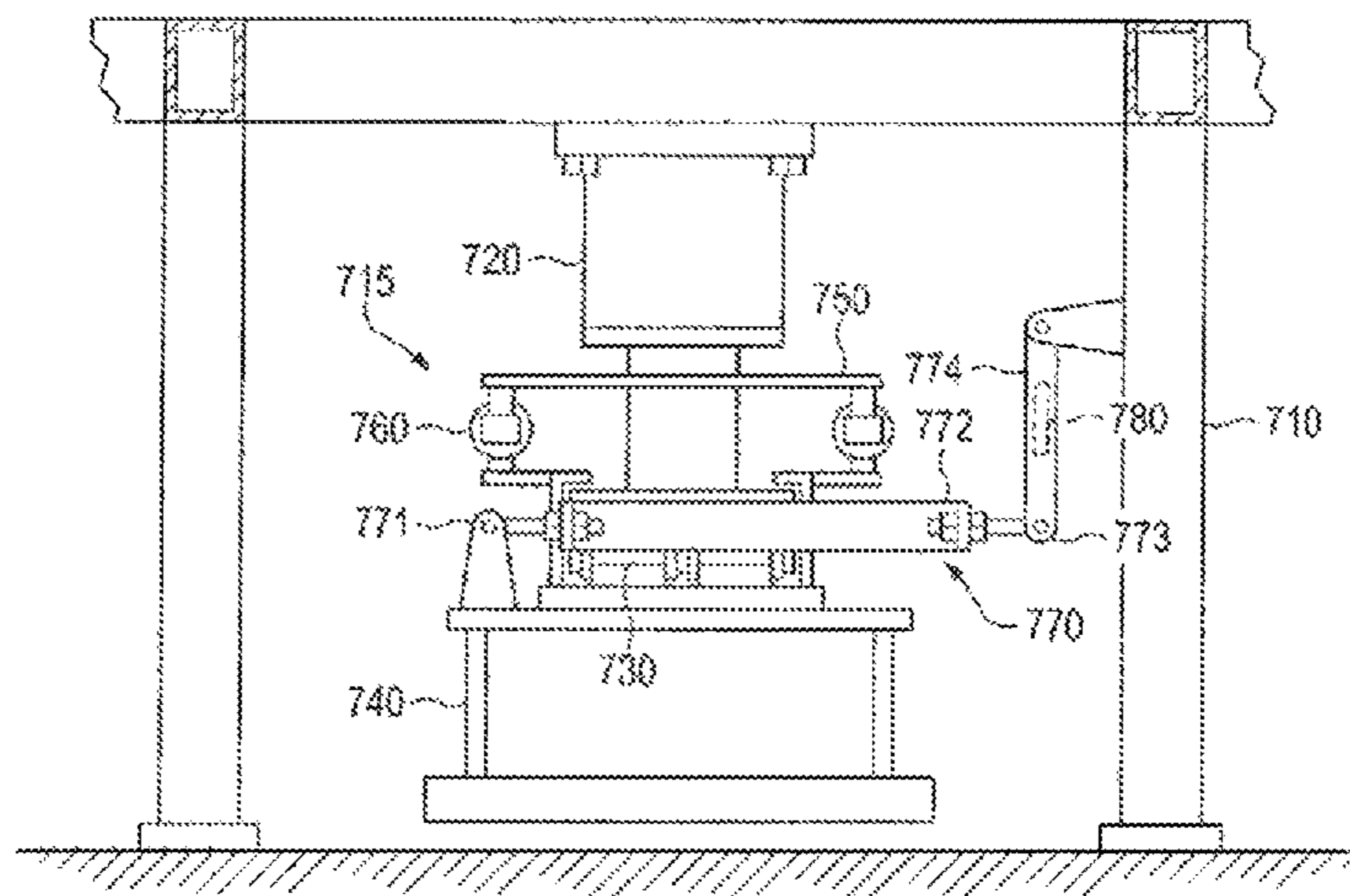
(Continued)

(57)

ABSTRACT

Embodiments of the present invention are directed to a load transporting apparatus that automatically aligns a support foot of the apparatus with a load-bearing frame connected to the load transporting apparatus during a recovery phase of an incremental walking movement. In particular, the load transporting apparatus includes a linking device attached to a support foot of the apparatus and a biasing device connected to the linking device that is deflected during non-linear load transporting movements, where the biasing device acts to automatically return the support foot to an aligned position relative to the load-bearing frame after a non-linear movement has been completed and the support foot is raised above a ground surface.

28 Claims, 17 Drawing Sheets



Related U.S. Application Data						
(60)	Provisional application No. 61/576,657, filed on Dec. 16, 2011.		RE29,541 E	2/1978	Russell	
			4,135,340 A	1/1979	Cox	
			4,206,935 A	6/1980	Sheppard et al.	
			4,252,204 A	2/1981	Bishop	
			4,290,495 A	9/1981	Eiliston	
(51)	Int. Cl.		4,296,820 A	10/1981	Loftis	
	<i>B62D 57/032</i> (2006.01)		4,324,077 A	4/1982	Woolslayer	
	<i>E21B 15/00</i> (2006.01)		4,324,302 A	4/1982	Rabinovitch	
			4,334,587 A	6/1982	Rangaswamy	
(58)	Field of Classification Search		4,371,041 A	2/1983	Becker	
	USPC 180/8.1, 8.5, 8.6		4,375,892 A	3/1983	Jenkins	
	See application file for complete search history.		4,405,019 A	9/1983	Frisbee	
			4,406,339 A	9/1983	Spencer	
			4,423,560 A	1/1984	Rivinius	
(56)	References Cited		4,489,954 A	12/1984	Yasui et al.	
	U.S. PATENT DOCUMENTS		4,491,449 A	1/1985	Hawkins	
			4,555,032 A	11/1985	Mick	
	1,289,207 A	12/1918	Leach	4,655,467 A	4/1987	Kitzmiller et al.
	1,429,551 A	9/1922	Anderson	4,759,414 A	7/1988	Willis
	1,615,055 A	1/1927	Turner	4,821,816 A	4/1989	Willis
	1,627,249 A	5/1927	Page	4,823,870 A	4/1989	Sorokan
	1,692,121 A	11/1928	Grant	4,831,795 A	5/1989	Sorokan
	1,879,446 A	2/1931	Page	4,842,298 A	6/1989	Jarvis
	1,914,692 A	6/1933	Kakaska	5,015,147 A	5/1991	Taylor
	1,915,134 A	6/1933	MacPherson	5,178,406 A	1/1993	Reynolds
	2,132,184 A	10/1937	Poche	5,245,882 A	9/1993	Kallenberger
	2,164,120 A	6/1939	Page	5,248,005 A	9/1993	Mochizuki
	2,247,782 A	7/1941	Martinson	5,398,396 A	3/1995	Sanders
	2,259,200 A	10/1941	Cameron et al.	5,492,436 A	2/1996	Suksumake
	2,290,118 A	7/1942	Page	5,575,346 A	11/1996	Yberle
	2,380,431 A	7/1945	Harding	5,600,905 A	2/1997	Kallenberger
	2,399,375 A	4/1946	Mullins	5,603,174 A	2/1997	Kallenberger
	2,452,632 A	11/1948	McLeod	5,613,444 A	3/1997	Ahmadian et al.
	2,541,496 A	2/1951	Busick, Jr.	5,749,596 A	5/1998	Jensen et al.
	2,616,677 A	11/1952	Compton	5,794,723 A	8/1998	Caneer
	2,660,253 A	11/1953	Davidson	5,921,336 A	7/1999	Reed
	2,660,449 A	11/1953	MacPherson	6,089,583 A	7/2000	Taipale
	2,692,770 A	10/1954	Nallinger et al.	6,202,774 B1 *	3/2001	Claassen et al. 180/8.5
	2,914,127 A	8/1955	Ricourd	6,203,247 B1	3/2001	Schellstede
	2,777,528 A	1/1957	Jourdain	6,345,831 B1	2/2002	deMarcellus
	2,901,240 A	8/1959	Fikse	6,474,926 B2	11/2002	Weiss
	2,935,309 A	5/1960	McCarthy	6,554,145 B1	4/2003	Fantuzzi
	2,942,676 A *	6/1960	Kraus 180/8.5	6,554,305 B2	4/2003	Fulks
	3,078,941 A	2/1963	Baron	6,581,525 B2 *	6/2003	Smith 104/96
	3,113,661 A	12/1963	Linke et al.	6,612,781 B1	9/2003	Jackson
	3,135,345 A	6/1964	Scruggs	6,651,991 B2	11/2003	Carlstedt et al.
	3,150,733 A	9/1964	Goebel	6,820,887 B1	11/2004	Riggle
	3,249,168 A	5/1966	Klein	6,857,483 B1	2/2005	Dirks
	3,255,836 A	6/1966	Hoppmann et al.	6,962,030 B2	11/2005	Conn
	3,265,145 A	8/1966	Beitzel	7,182,163 B1	2/2007	Gipson
	3,334,849 A	8/1967	Bronder	7,308,953 B2	12/2007	Barnes
	3,362,553 A	1/1968	Weinmann	7,357,616 B2	4/2008	Andrews
	3,375,892 A	4/1968	Kraschnewski et al.	7,681,674 B1	3/2010	Barnes et al.
	3,446,301 A	5/1969	Thomas	7,806,207 B1	10/2010	Barnes et al.
	3,448,994 A	6/1969	King et al.	7,819,209 B1	10/2010	Bezner
	3,490,786 A	1/1970	Ravenel	7,882,915 B1 *	2/2011	Wishart 180/203
	3,493,064 A	2/1970	Wilson	8,019,472 B2	9/2011	Montero
	3,512,597 A *	5/1970	Baron 180/8.5	8,051,930 B1	11/2011	Barnes et al.
	3,527,313 A	9/1970	Reimann	8,250,816 B2	8/2012	Donnally
	3,528,341 A	9/1970	Rieschel	8,468,753 B2	6/2013	Donnally
	3,576,225 A *	4/1971	Chambers 180/8.5	8,490,724 B2 *	7/2013	Smith et al. B62D 57/02 180/8.6
	3,612,201 A	10/1971	Smith	8,490,727 B2	7/2013	Smith et al.
	3,638,747 A *	2/1972	Althoff 180/8.5	8,556,003 B2	10/2013	Soucek
	3,674,103 A	7/1972	Kiekhaefer	8,561,733 B2 *	10/2013	Smith et al. B62D 57/02 180/8.1
	3,734,220 A	5/1973	Smith	8,573,334 B2 *	11/2013	Smith et al. 180/8.5
	3,754,361 A	8/1973	Branham	8,646,549 B2	2/2014	Ledbetter
	3,765,499 A	10/1973	Harmala	8,646,976 B2	2/2014	Stoik
	3,767,064 A	10/1973	Lutz	8,839,892 B2 *	9/2014	Smith et al. B62D 57/02 180/8.6
	3,767,224 A	10/1973	Schneeweiss			
	3,796,276 A	3/1974	Maeda			
	3,807,519 A *	4/1974	Patch 180/8.5	8,887,800 B2	11/2014	Havinga
	3,853,196 A *	12/1974	Guest 180/8.5	9,004,203 B2	4/2015	Smith
	3,866,425 A	2/1975	Morrice	9,045,178 B2	6/2015	Smith
	3,921,739 A *	11/1975	Rich et al. 180/8.5	9,463,833 B2	10/2016	Smith et al.
	3,951,225 A	4/1976	Schewenk	2004/0211598 A1	10/2004	Palidis
	4,014,399 A *	3/1977	Ruder 180/8.5	2004/0240973 A1	12/2004	Andrews
	4,021,978 A	5/1977	Busse	2006/0027373 A1	2/2006	Carriere
	4,048,936 A	9/1977	Uchizono	2006/0213653 A1	9/2006	Cunningham

(56)

References Cited

OTHER PUBLICATIONS

U.S. PATENT DOCUMENTS

2009/0000218	A1	1/2009	Lee	
2009/0188677	A1	7/2009	Ditta	
2009/0200856	A1	8/2009	Cehade	
2009/0283324	A1	11/2009	Konduc	
2010/0252395	A1	10/2010	Lehtonen	
2011/0072737	A1	3/2011	Wasterval	
2011/0114386	A1	5/2011	Souчек	
2012/0219242	A1	8/2012	Stoik	
2013/0153309	A1	6/2013	Smith et al.	
2013/0156538	A1	6/2013	Smith et al.	
2013/0156539	A1	6/2013	Smith et al.	
2013/0277124	A1	10/2013	Smith et al.	
2014/0014417	A1*	1/2014	Smith et al.	180/8.1
2014/0054097	A1*	2/2014	Bryant et al.	180/8.1
2014/0158342	A1	6/2014	Smith	
2014/0161581	A1*	6/2014	Smith et al.	414/800
2017/0021880	A1	1/2017	Smith	

FOREIGN PATENT DOCUMENTS

DE	2418411	10/1975
DE	4107314	9/1992
EP	469182	10/1990
EP	469182	2/1992
GB	2315464	2/1998
WO	2004103807	12/2004
WO	2006100166	9/2006
WO	2010136713	12/2010

Defendants' Second Amended Answer, Affirmative Defenses, and Counterclaims to Plaintiff's Complaint for Patent Infringement, Jul. 12, 2016, p. 9, Sections 15-18.

Defendants' First Amended Answer, Affirmative Defenses, and Counterclaims to Plaintiff's Complaint for Patent Infringement, Jun. 27, 2016, p. 9, Section 16.

Defendants' Answer, Affirmative Defenses and Counterclaims to Plaintiff's Complaint for Patent Infringement, Jun. 6, 2016, p. 6, Section 2.

Entro Industries, Inc. brochure "The Future of Rig Walkers", Jun. 2012; 4 pages.

Columbia Industries, LLC brochure "Kodiak Cub Rig Walking System", 2009; 4 pages.

Schwabe Williamson & Wyatt, PC "Listing of Related Cases", dated Jun. 28, 2016; 1 page.

Defendant Hydraulic Systems, Inc.'s Preliminary Invalidity Contentions, Nov. 18, 2016, pp. 1-6.

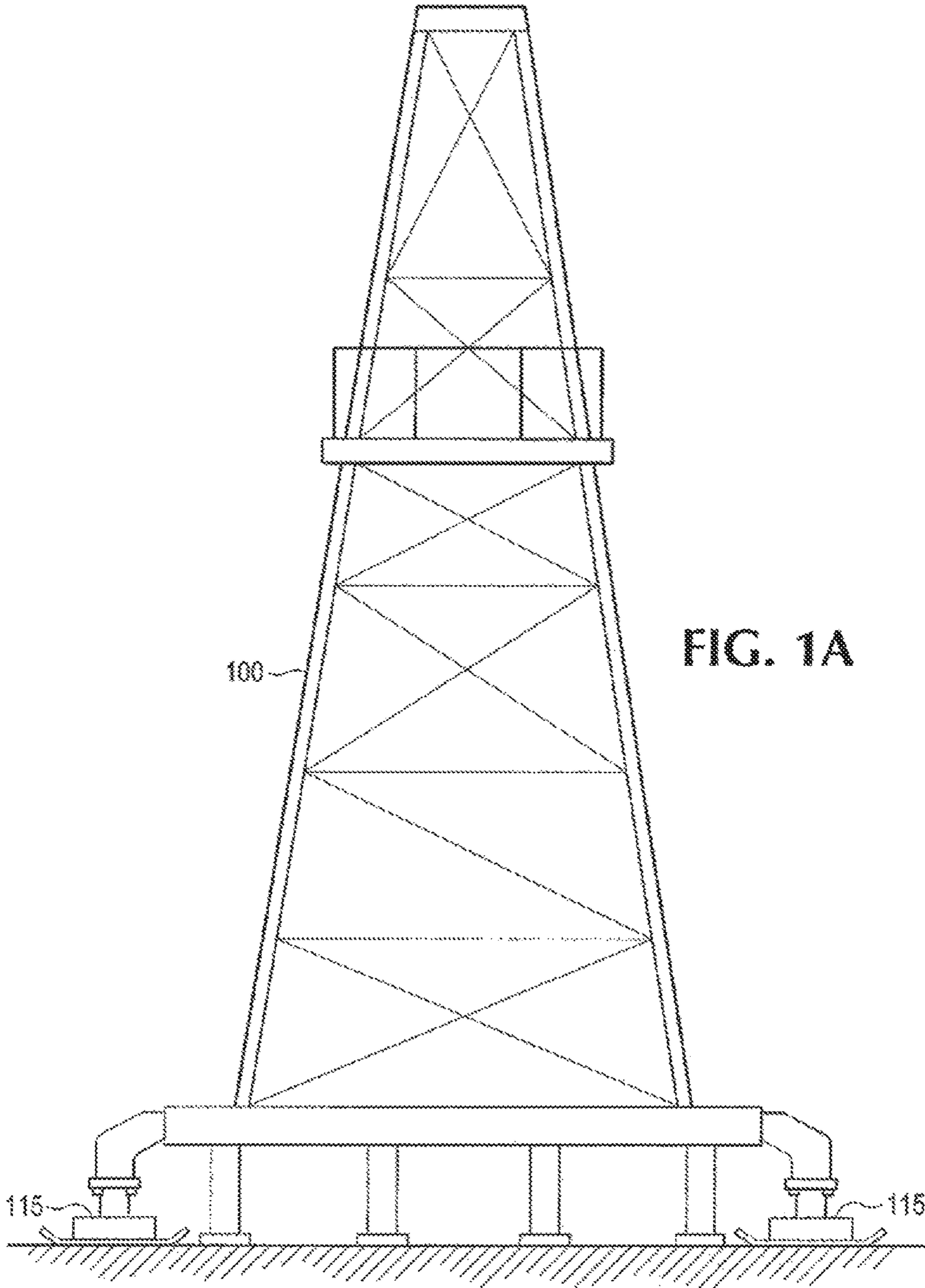
Defendant Hydraulic Systems, Inc.'s Supplemental Preliminary Invalidity Contentions, May 26, 2017, pp. 1-6.

Colby, Col. Joseph M., "Torsion-Bar Suspension", SAE Quarterly Transactions, vol. 2, No. 2, pp. 195-200, Apr. 1948.

Airstream Inc., "Airstream's New Dura-Torque Axle" Pamphlet.

Stolowitz Ford Cowger L.L.P., "Listing of Related Cases", Sep. 16, 2013, 1 page.

* cited by examiner



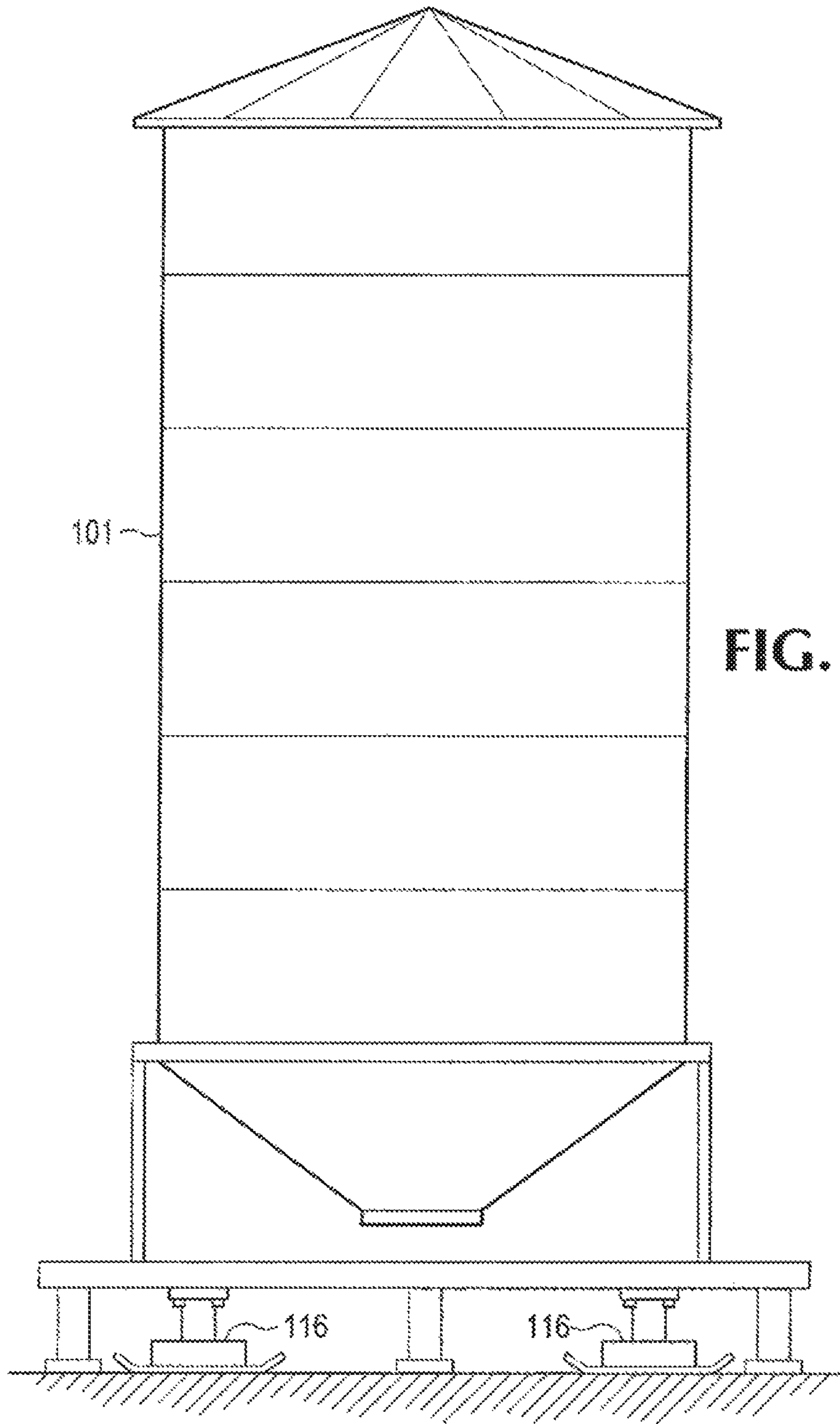


FIG. 1B

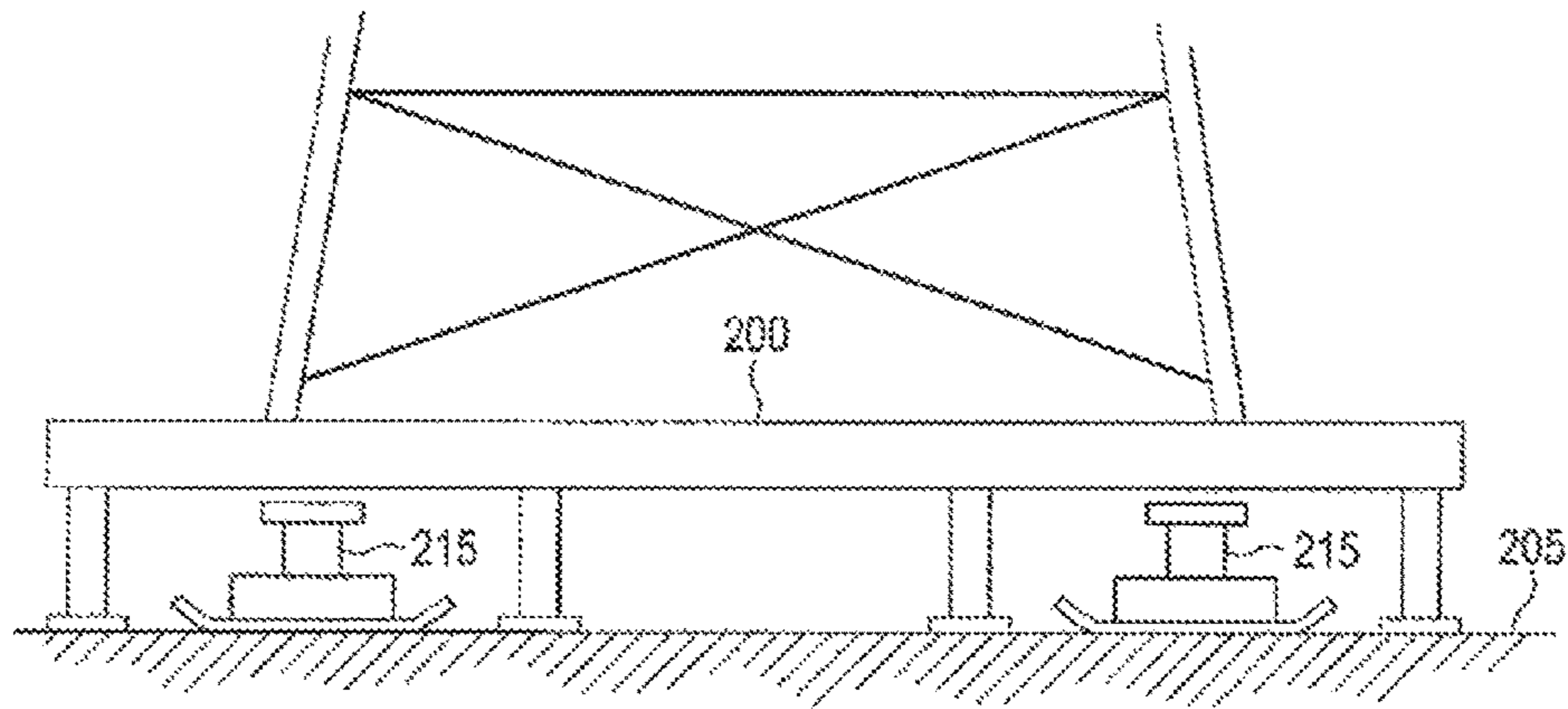


FIG. 2A

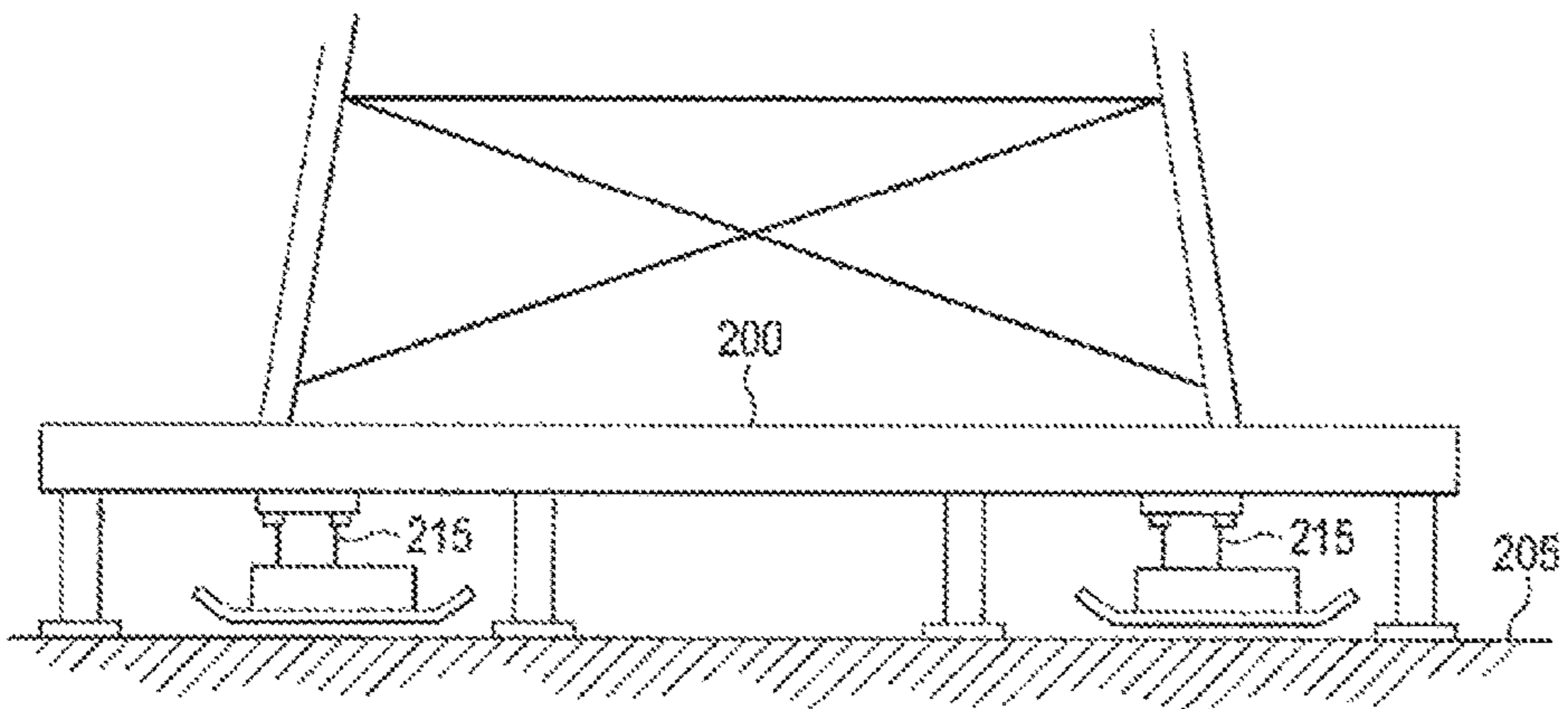


FIG. 2B

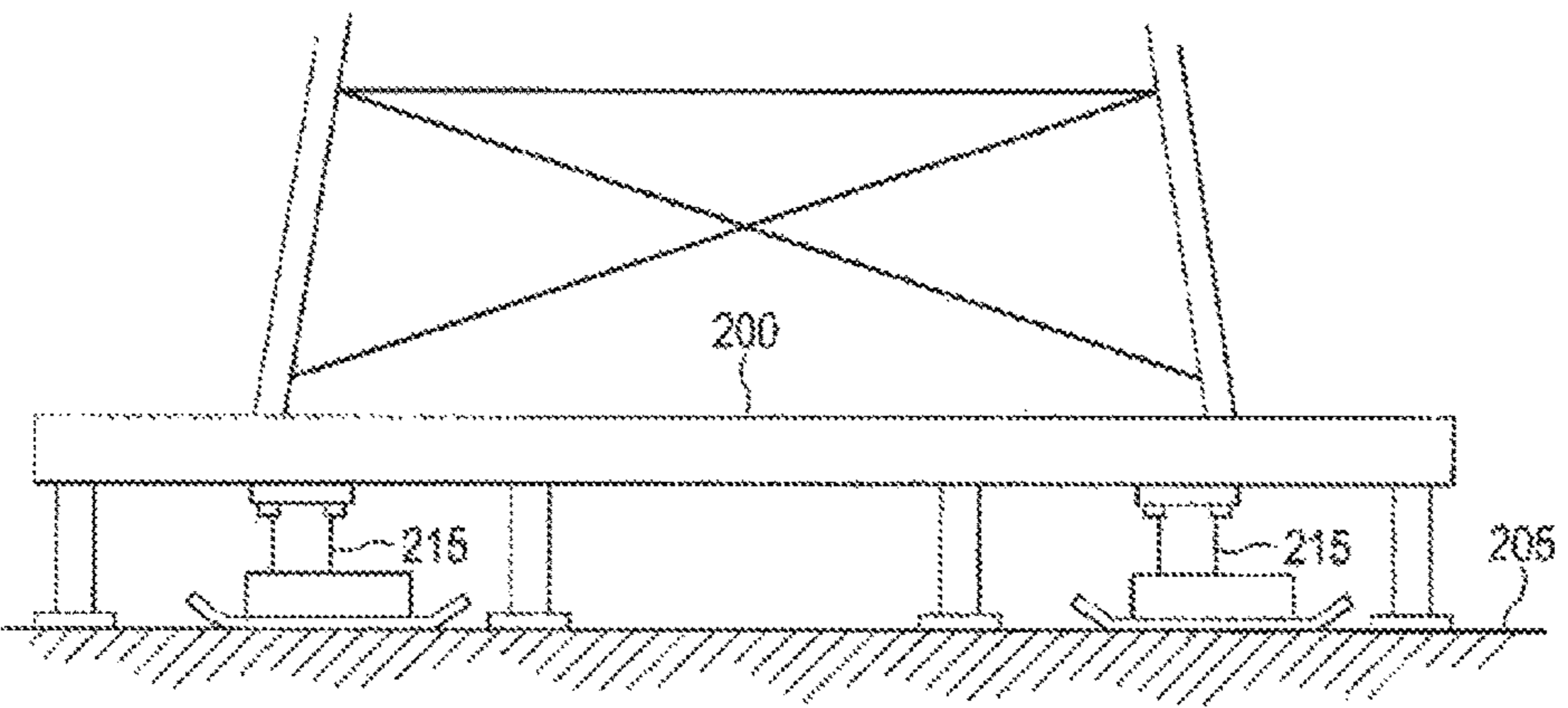


FIG. 2C

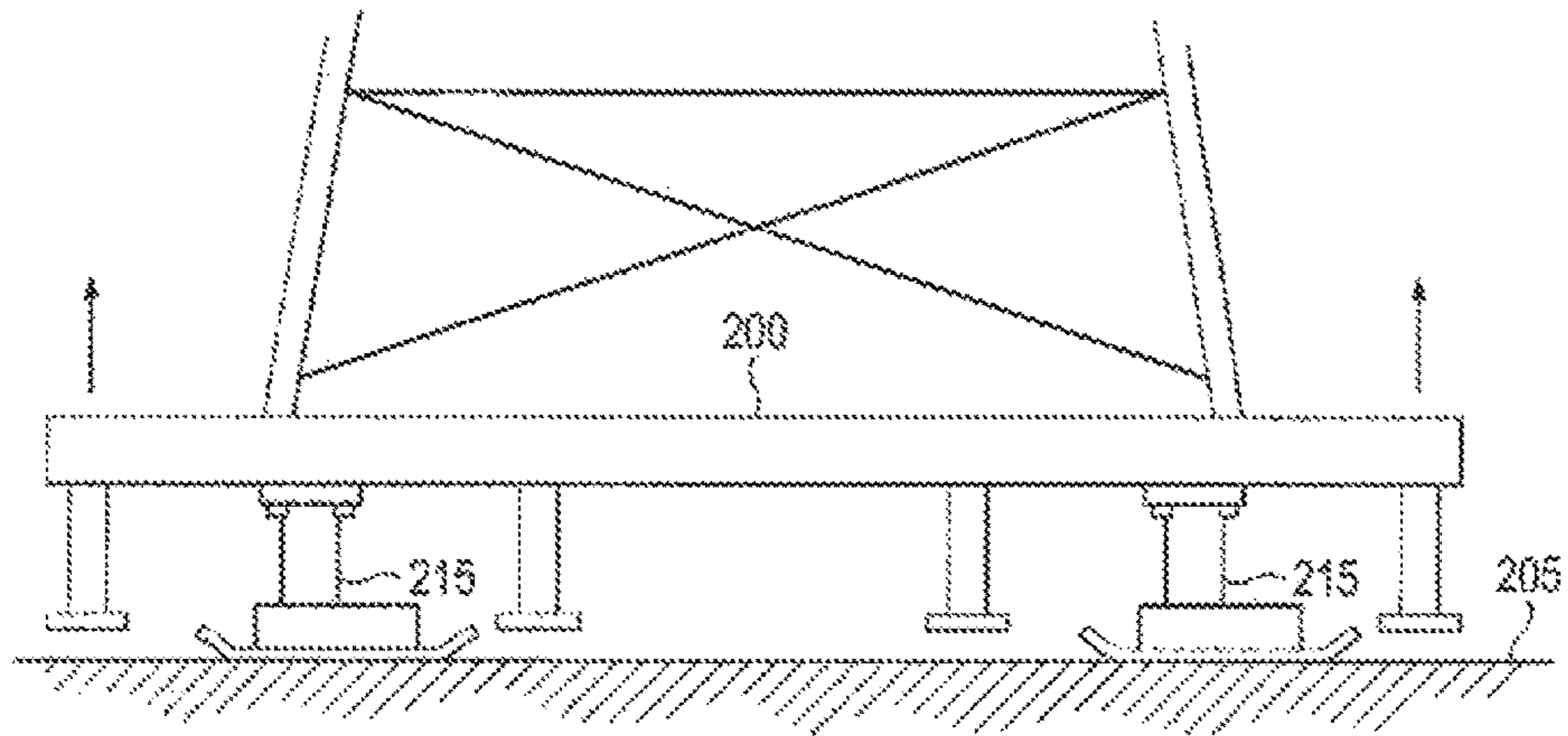


FIG. 2D

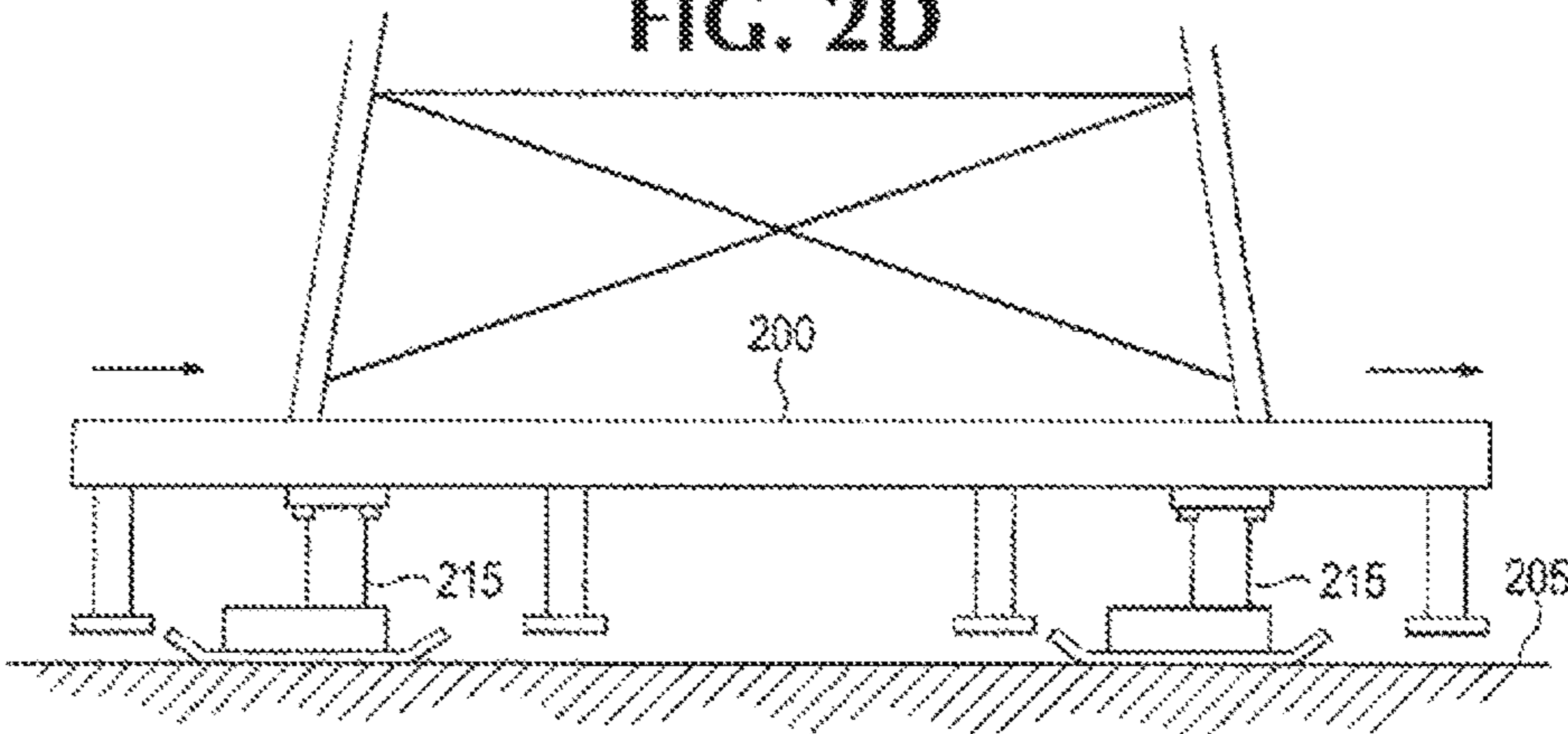


FIG. 2E

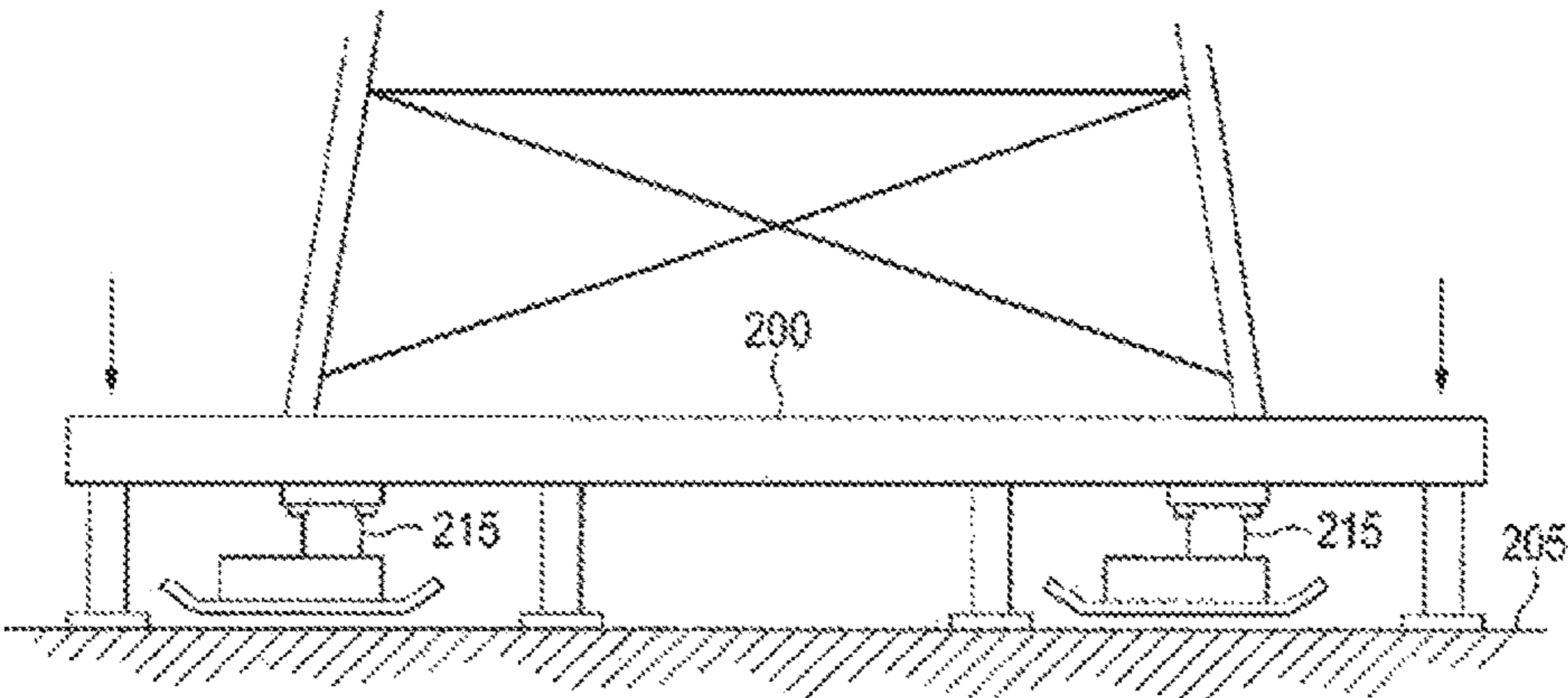


FIG. 2F

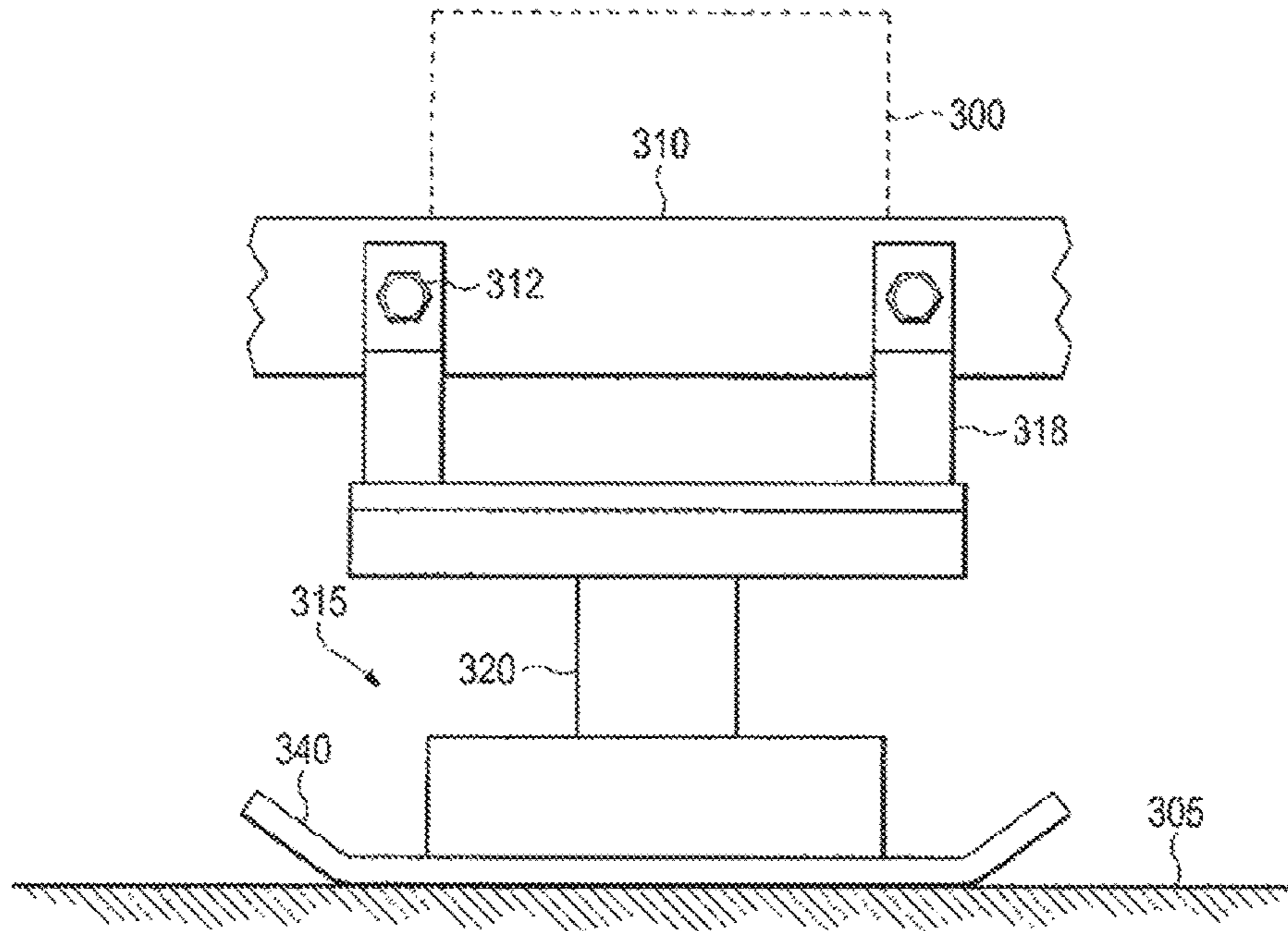


FIG. 3A

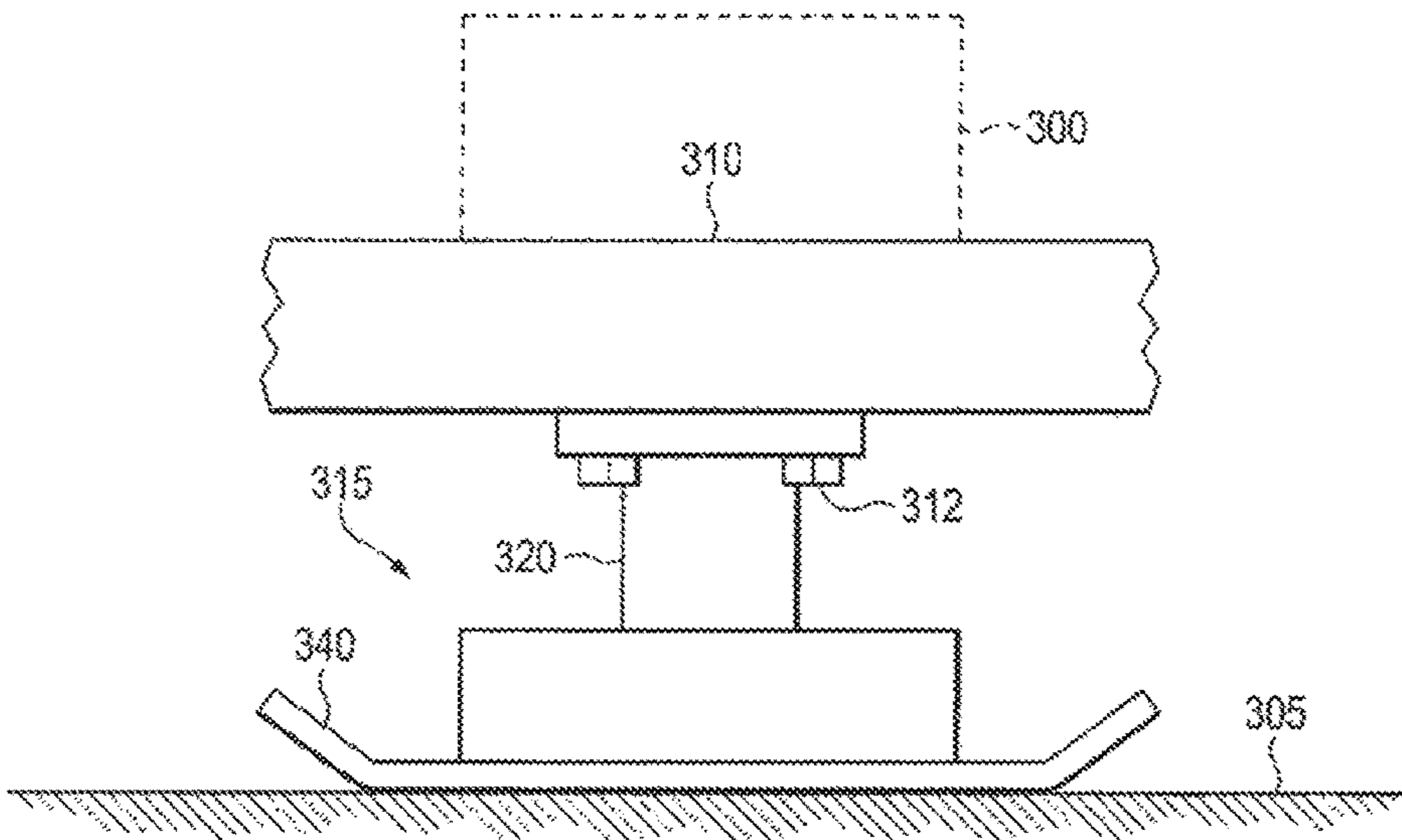


FIG. 3B

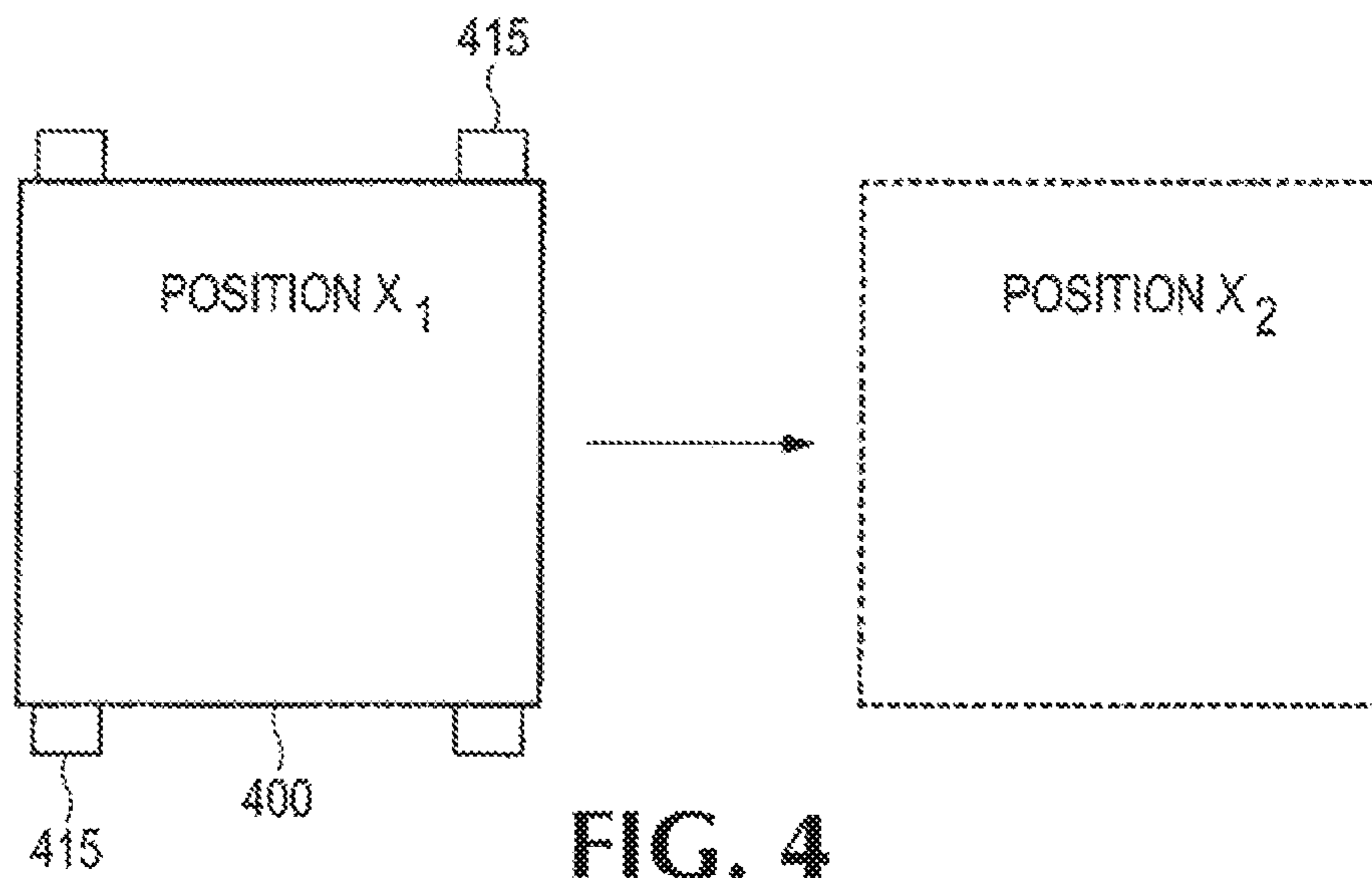


FIG. 4

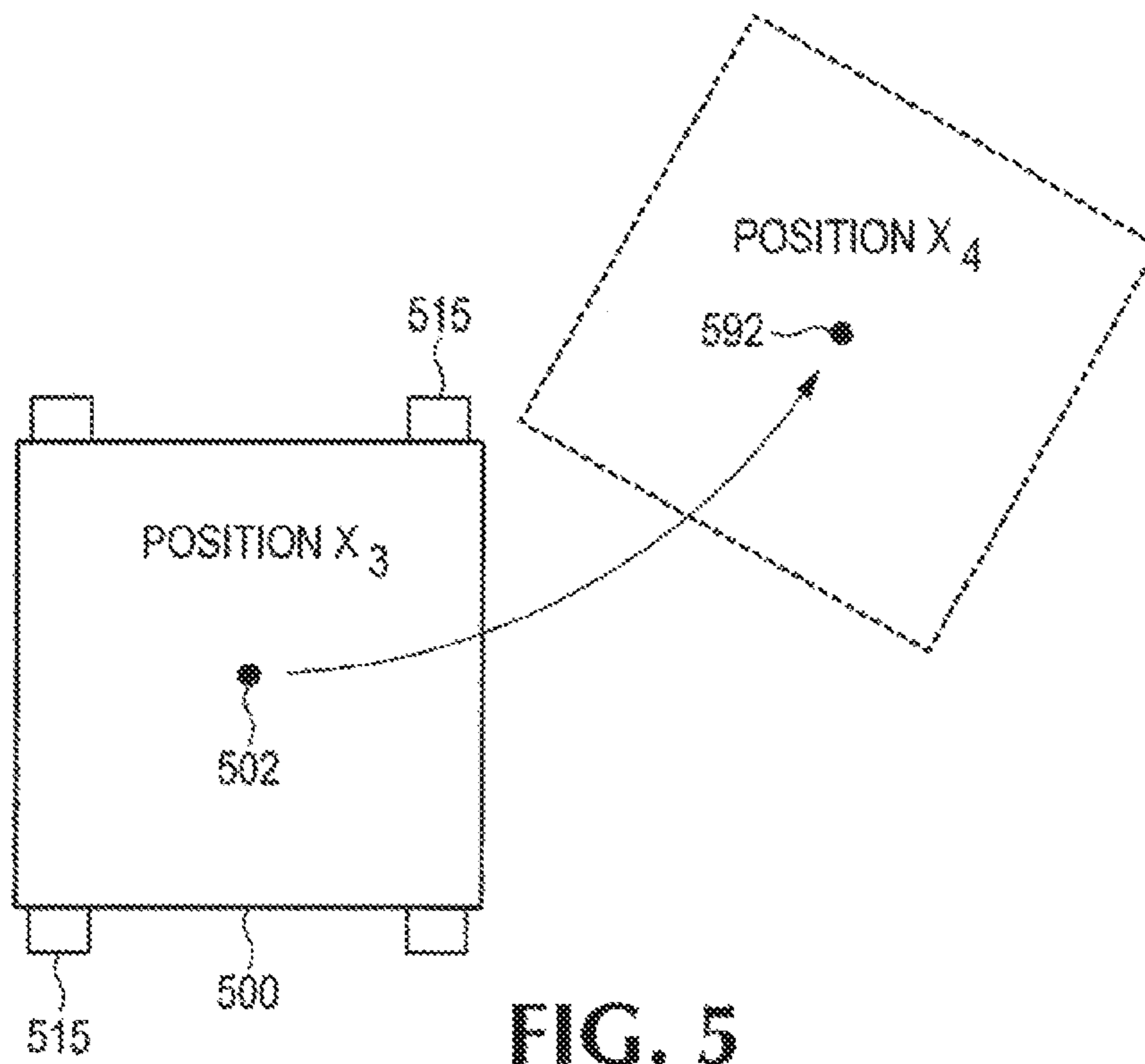


FIG. 5

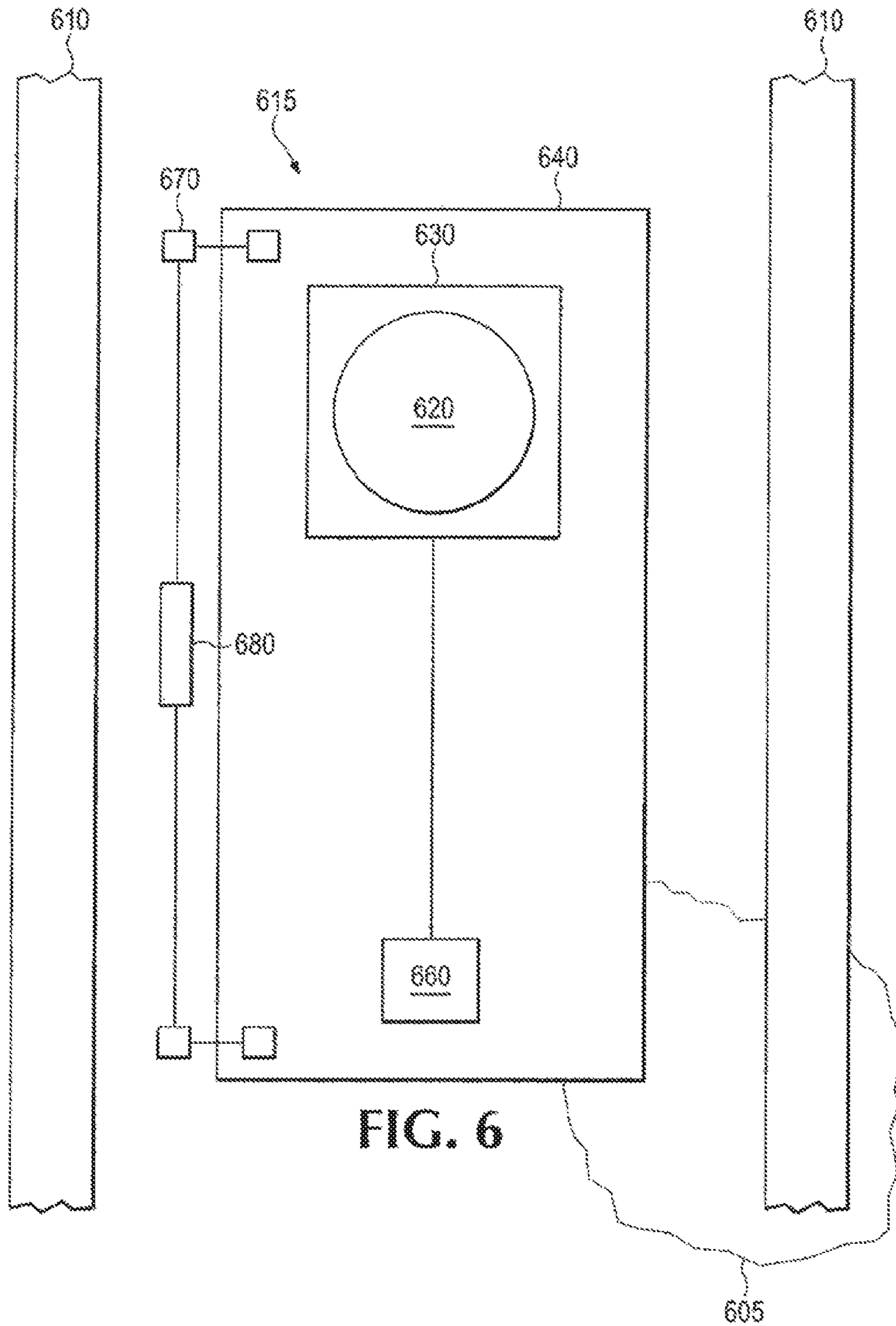


FIG. 6

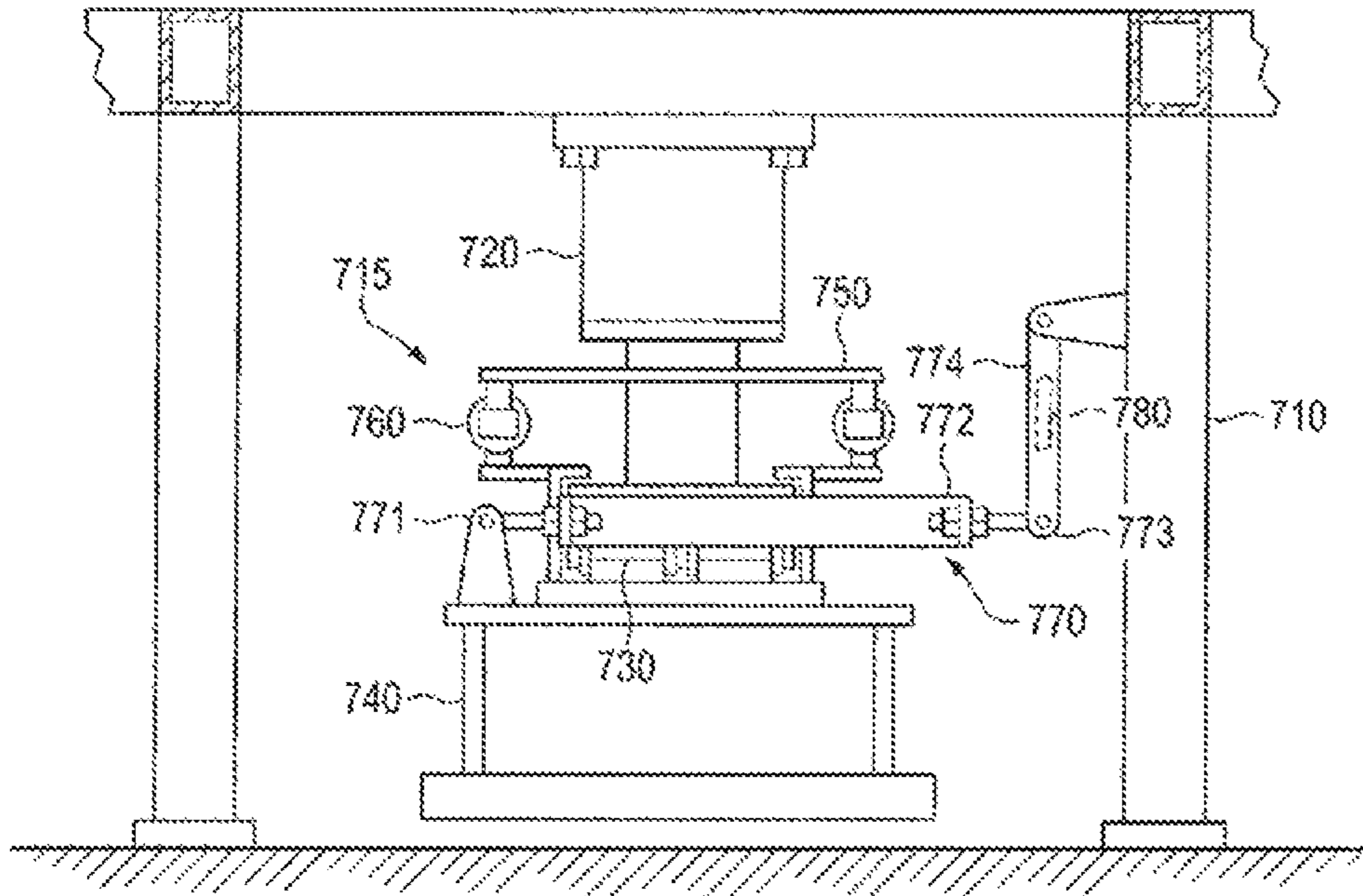


FIG. 7A

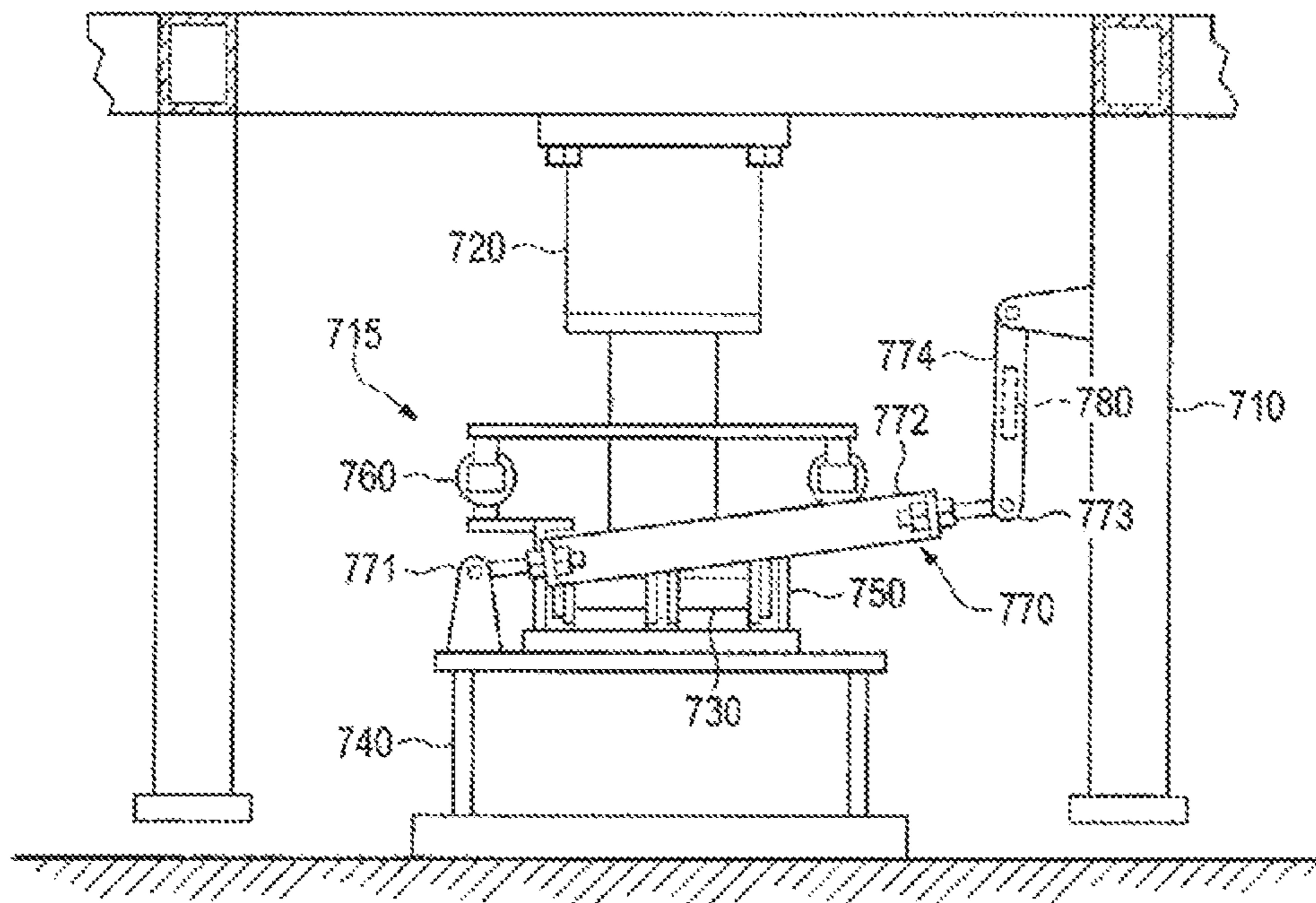


FIG. 7B

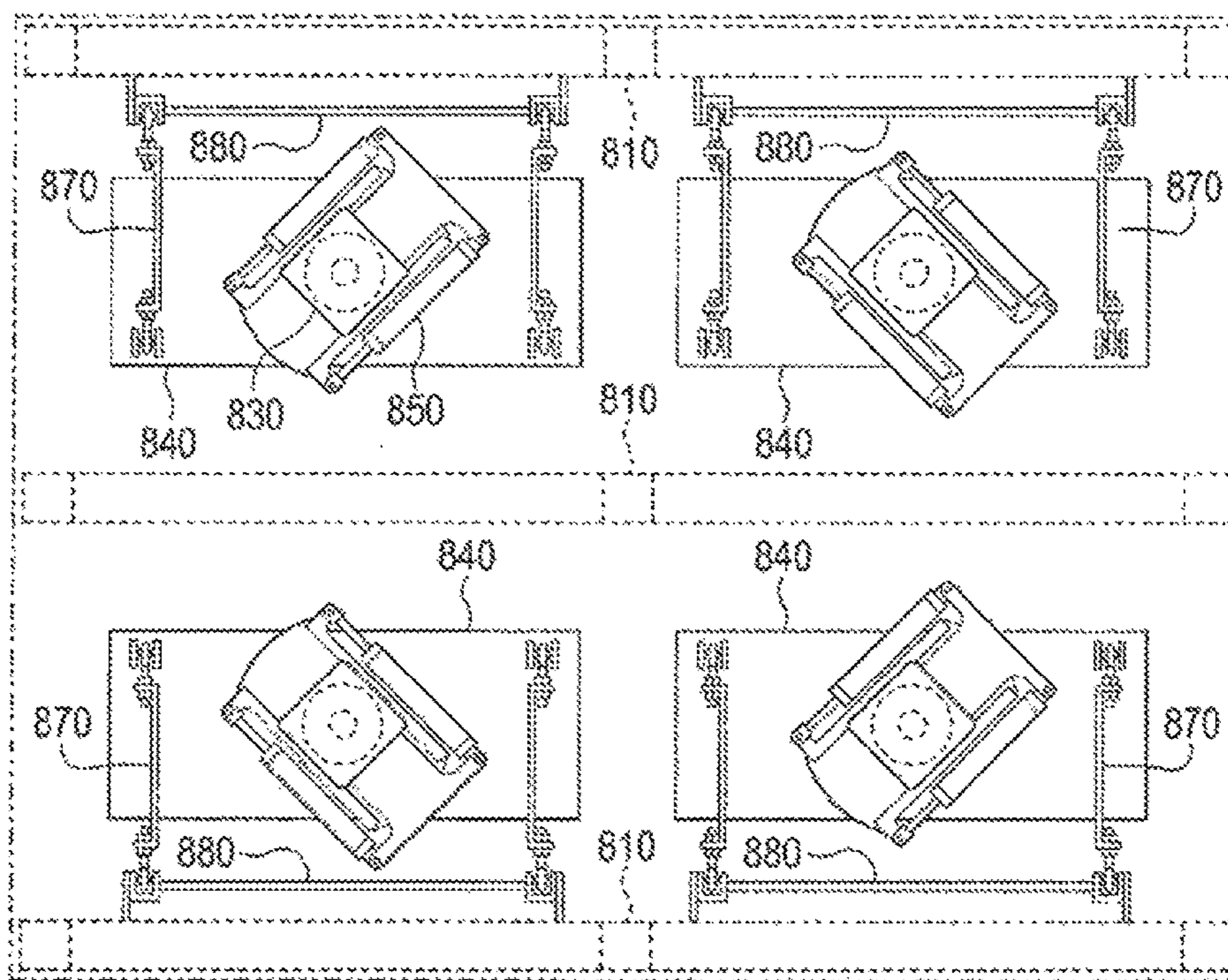
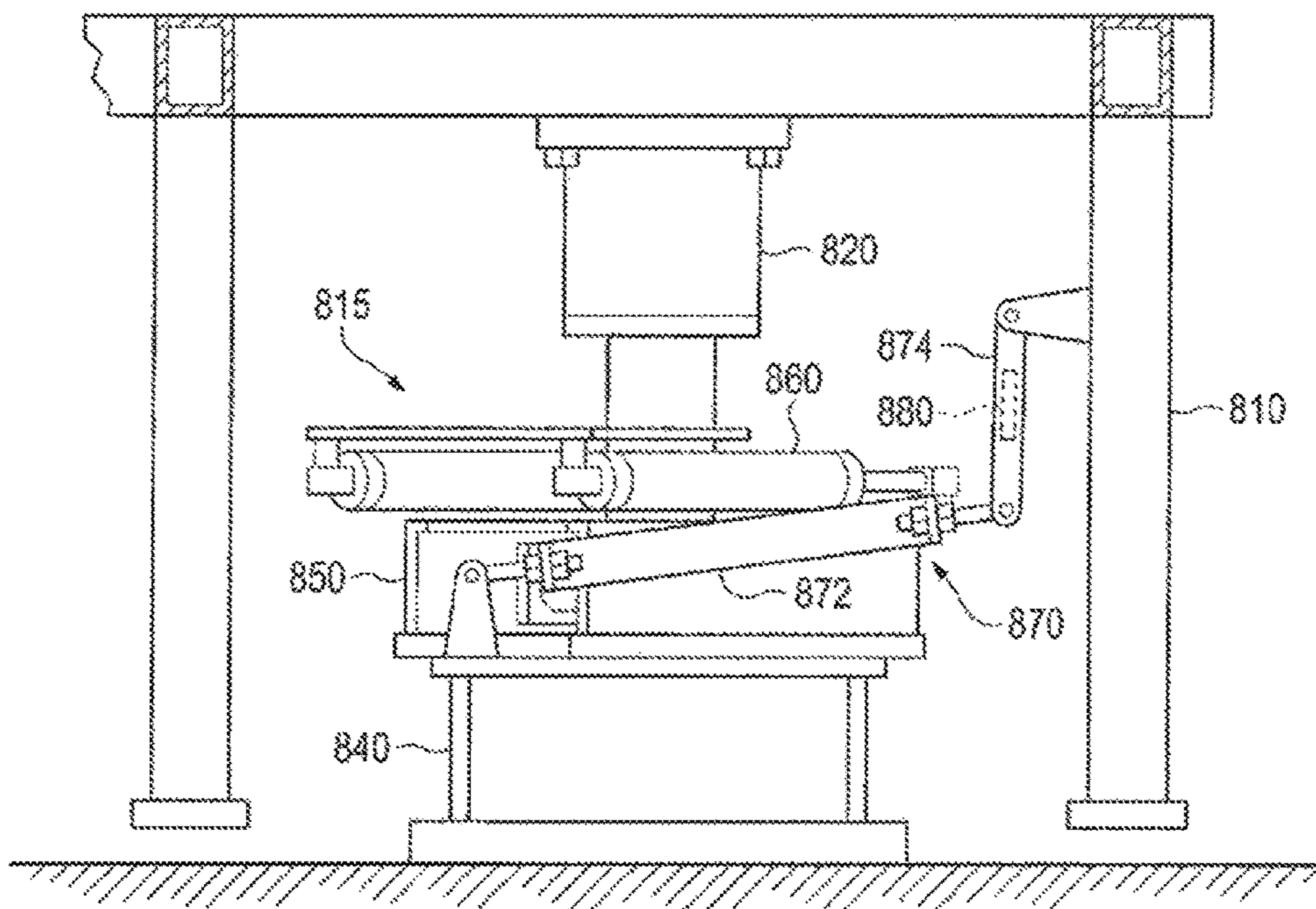


FIG. 8A

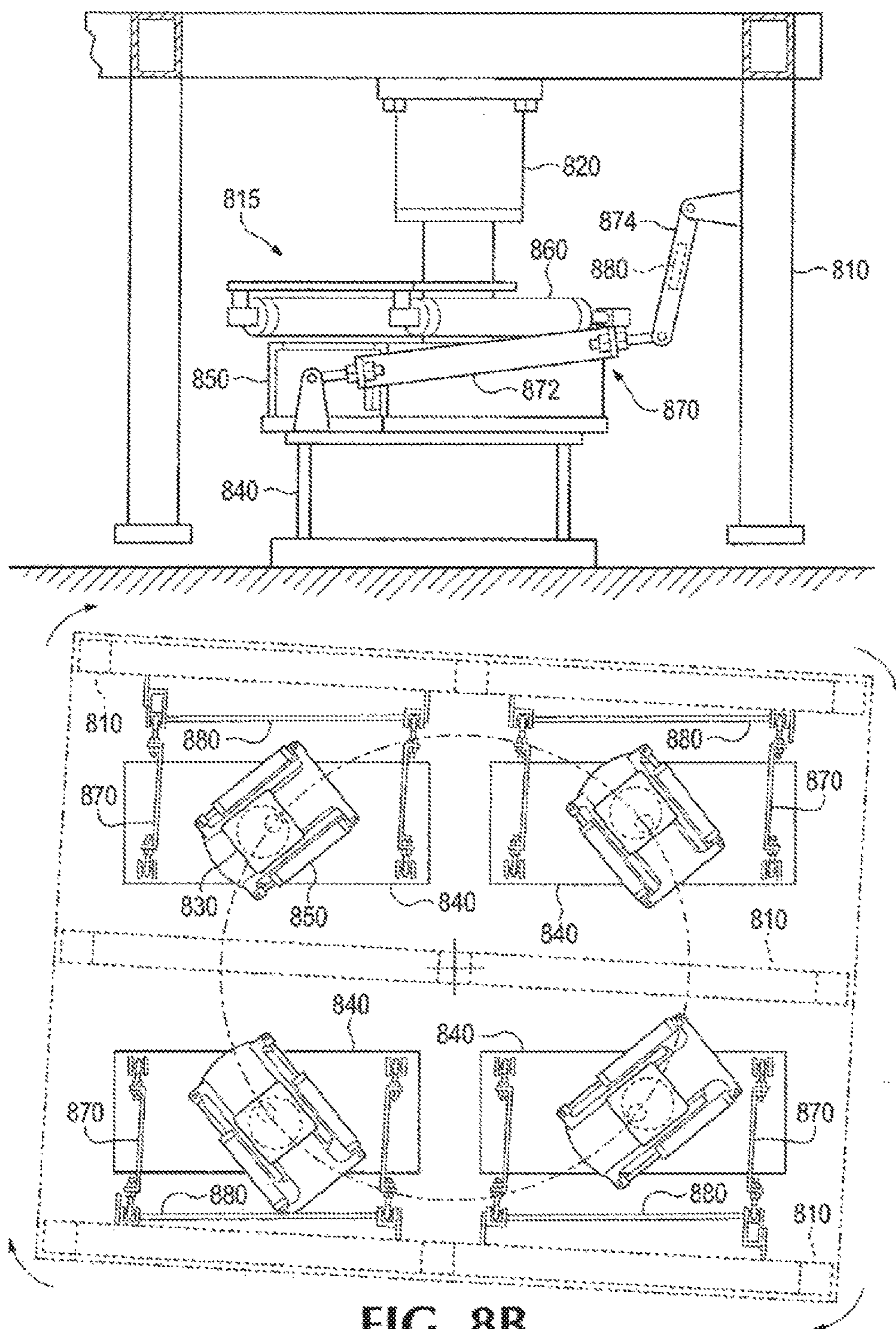


FIG. 8B

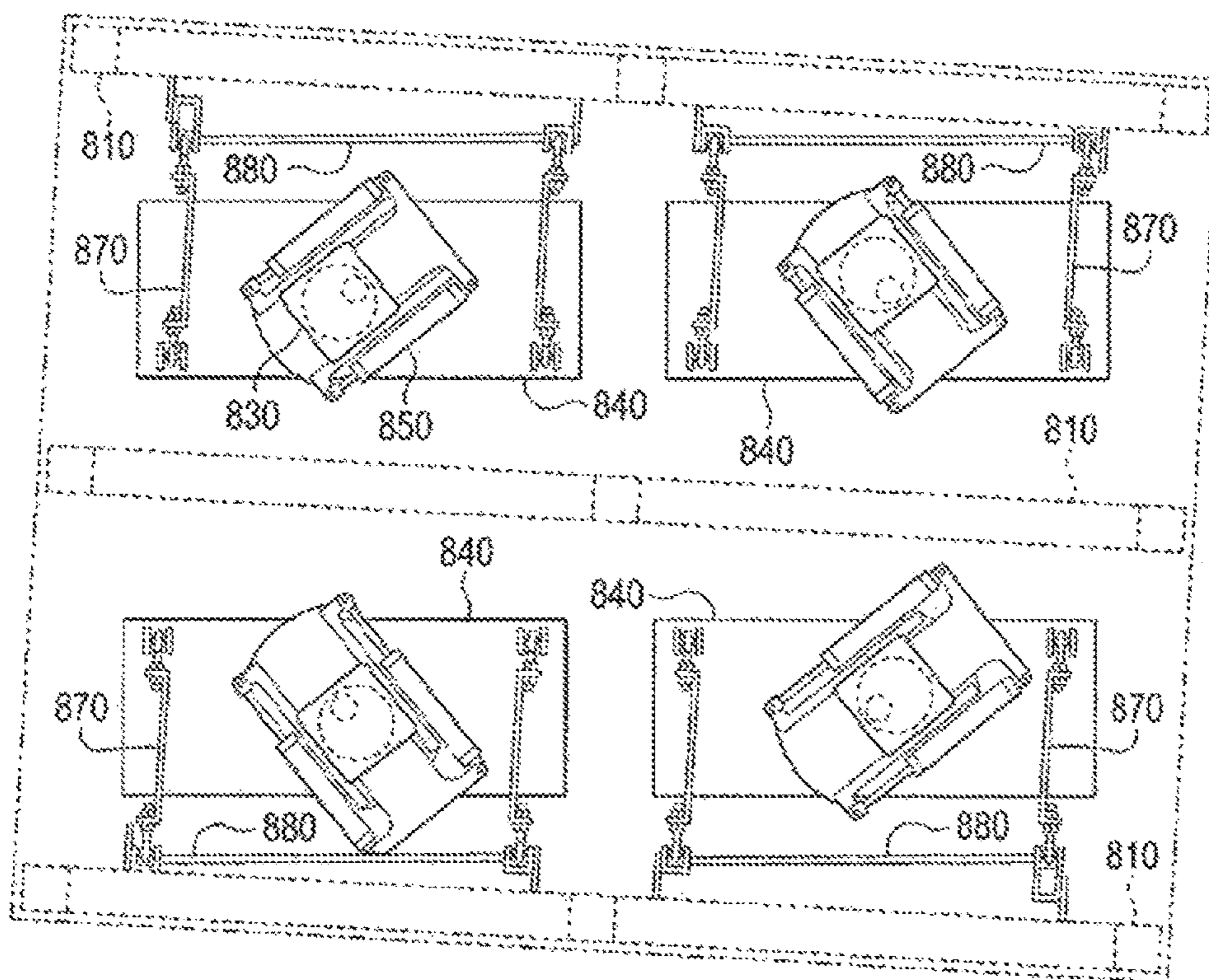
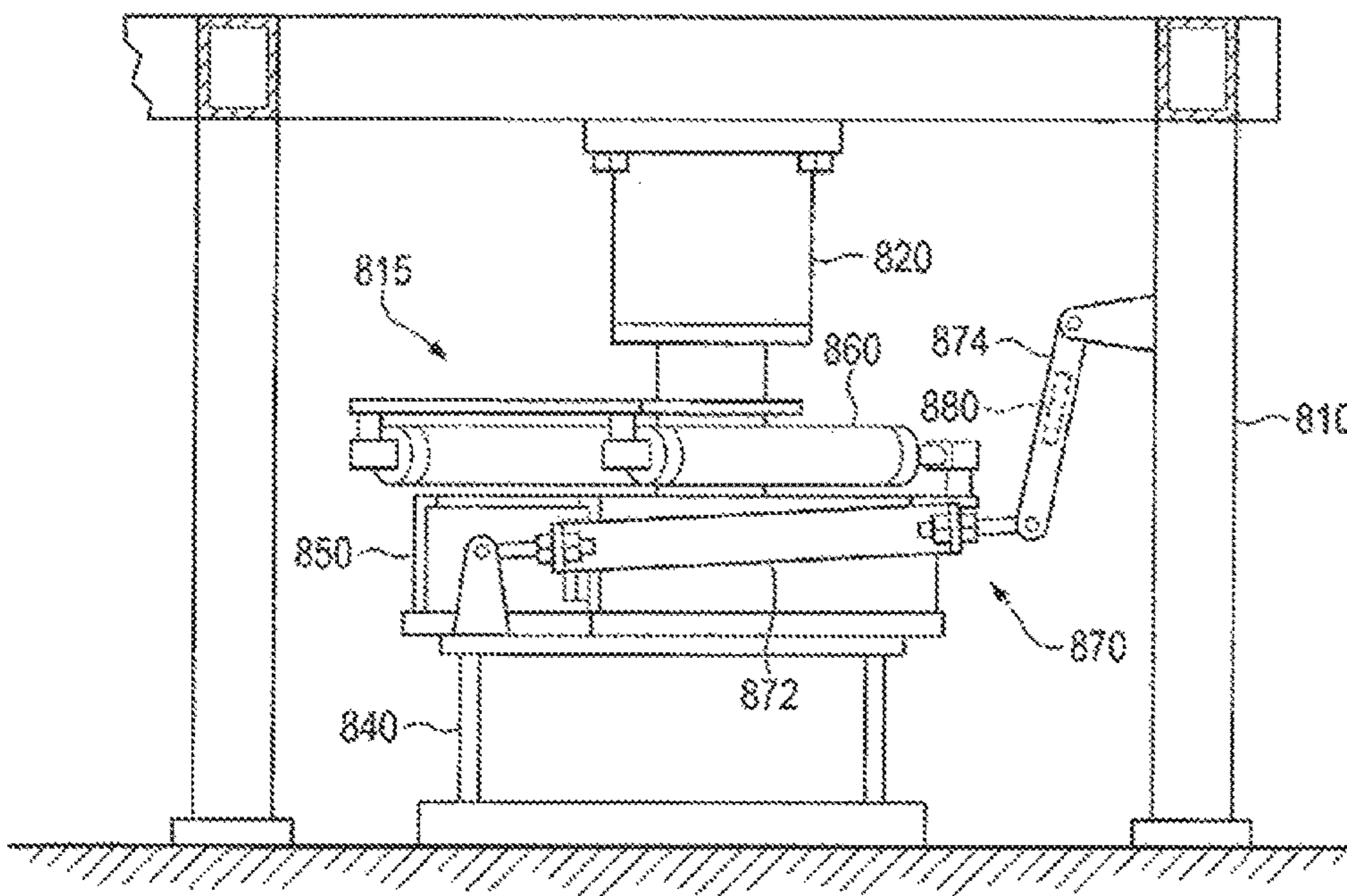


FIG. 8C

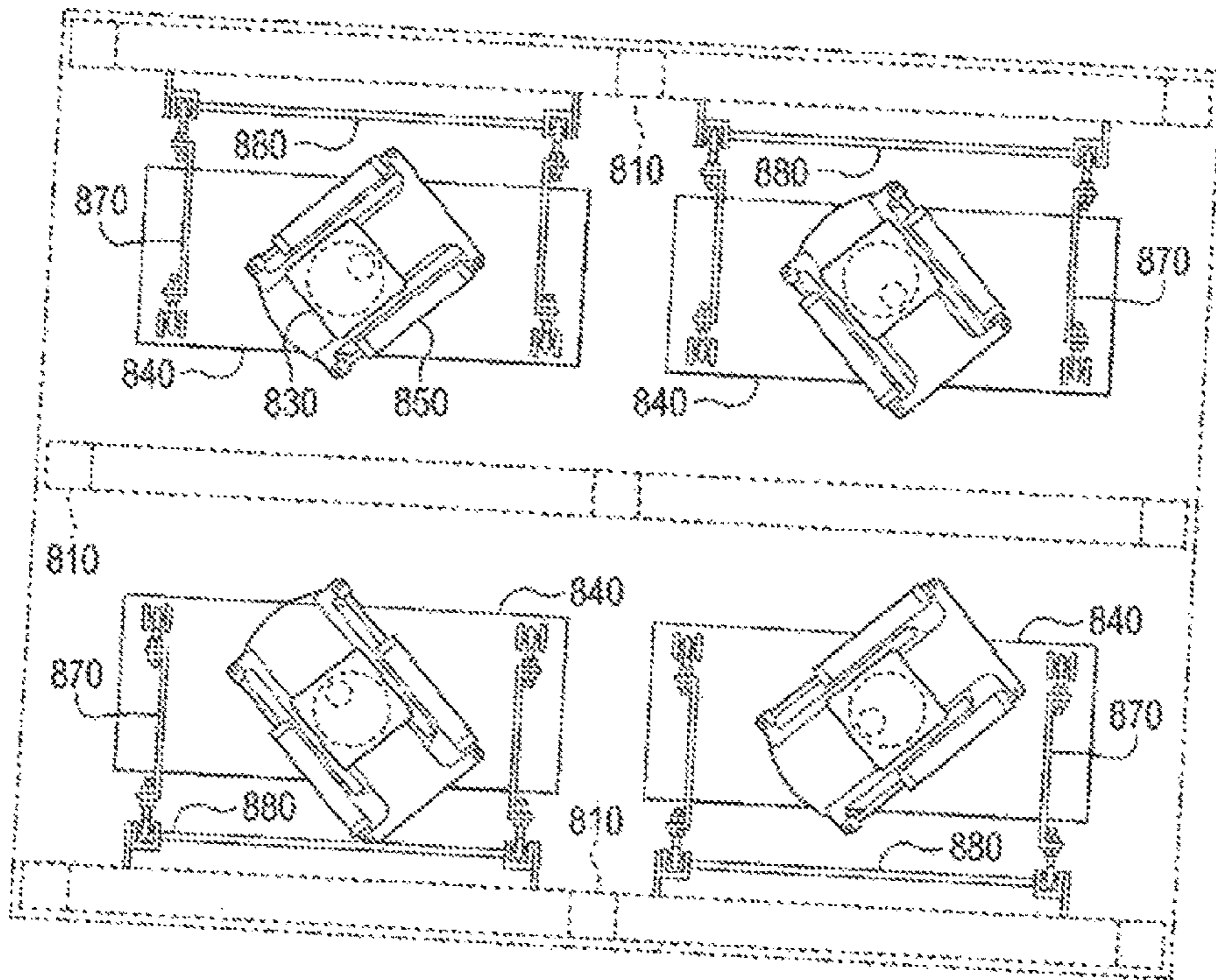
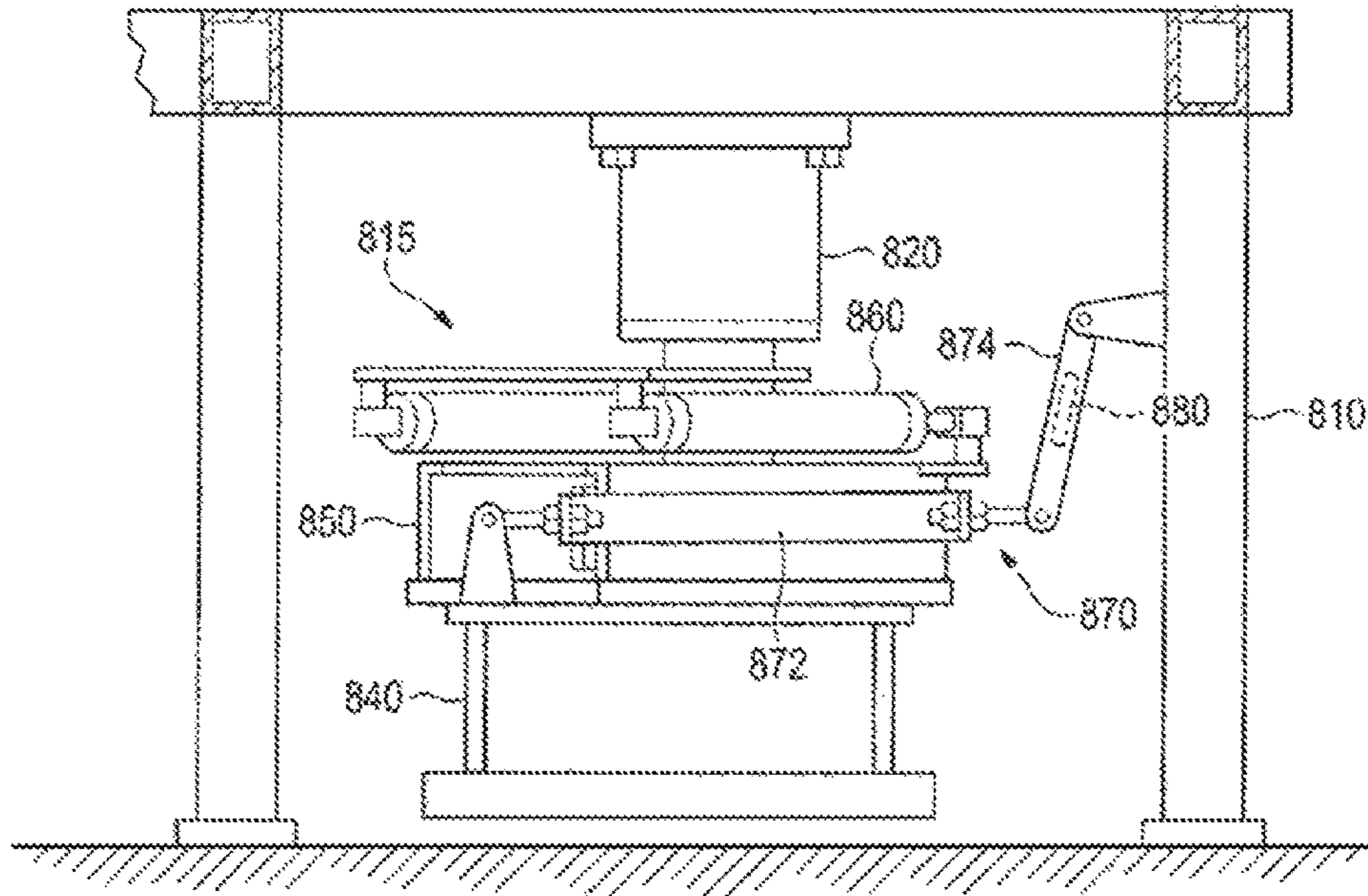


FIG. 8D

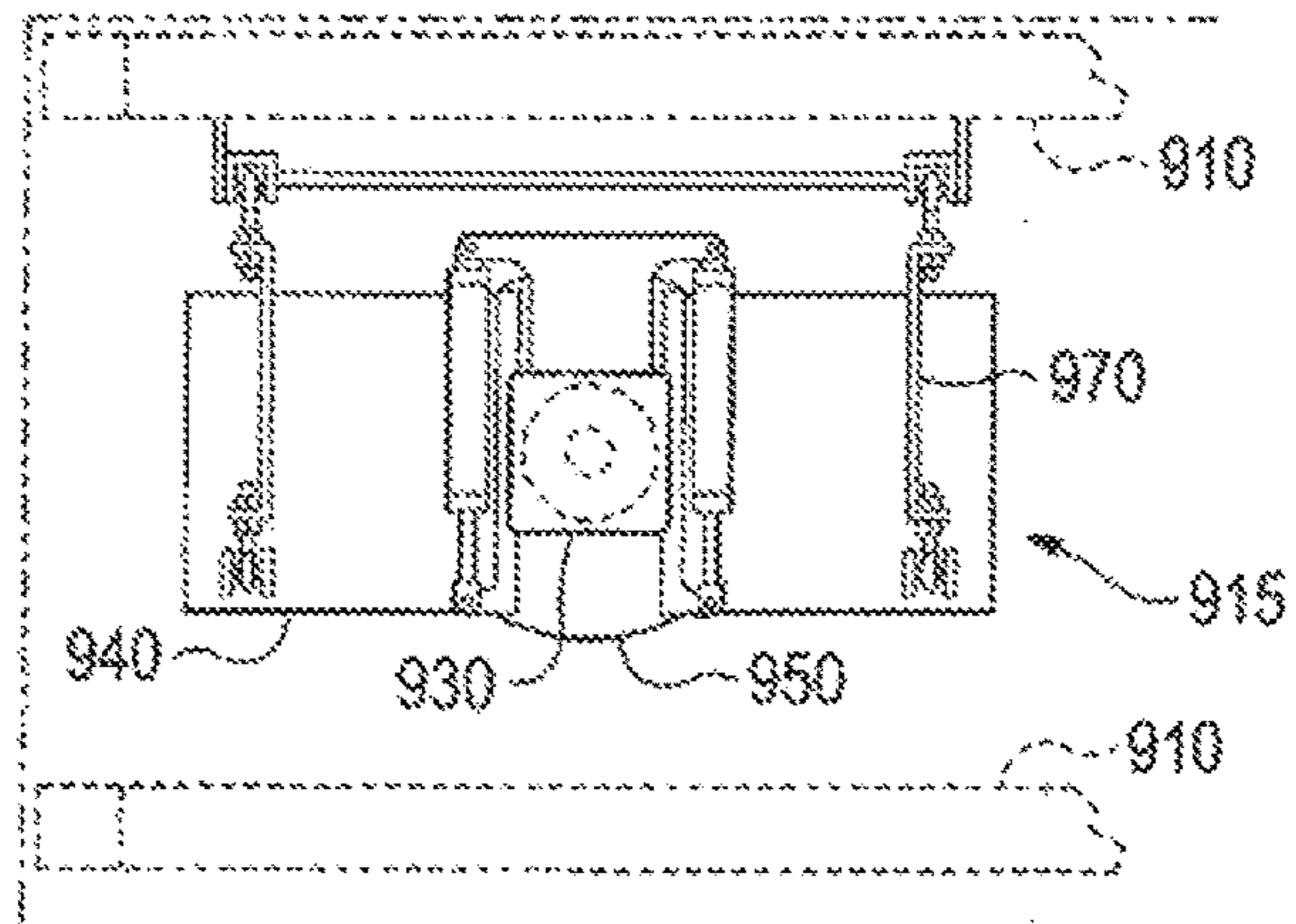


FIG. 9A

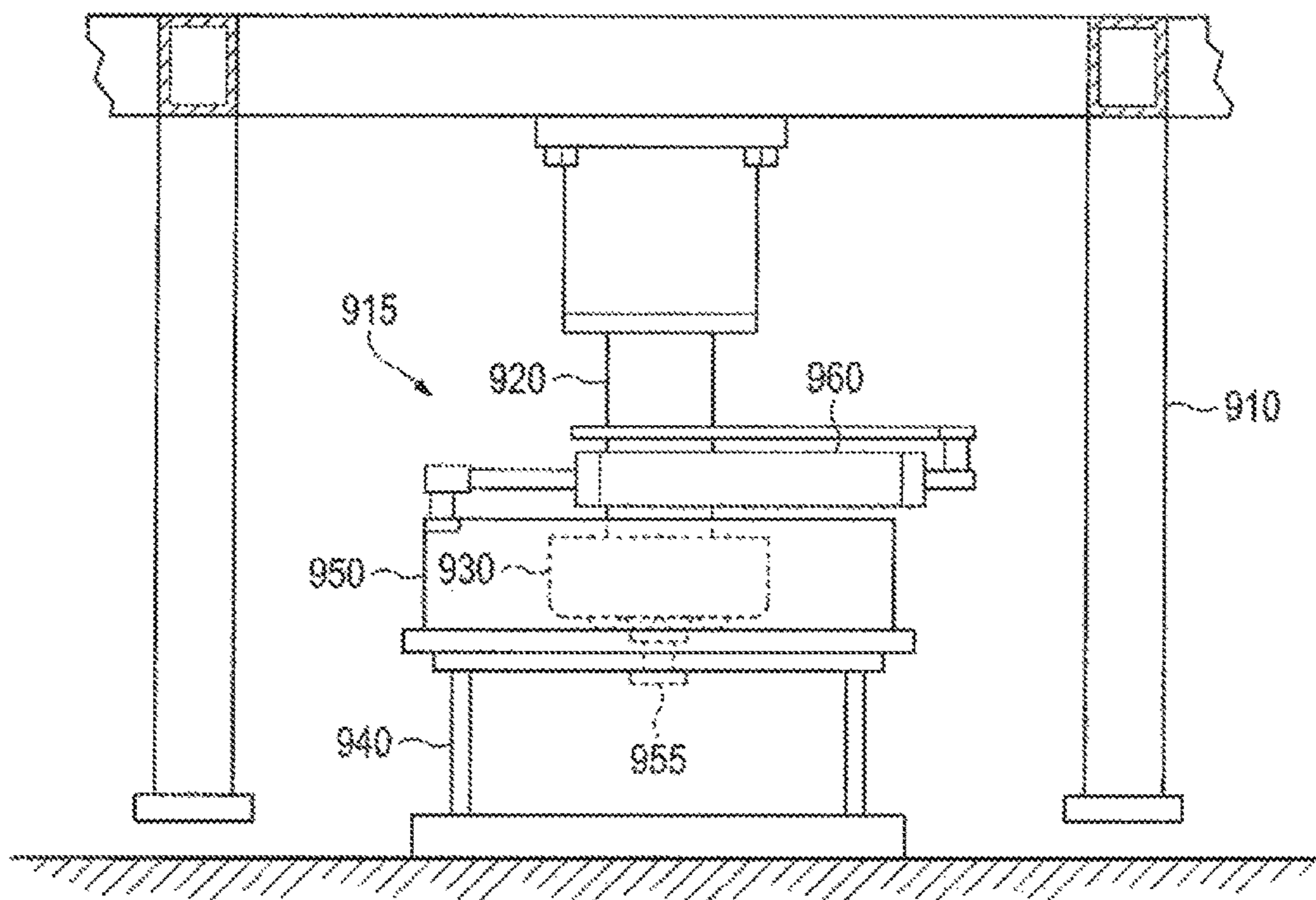


FIG. 9B

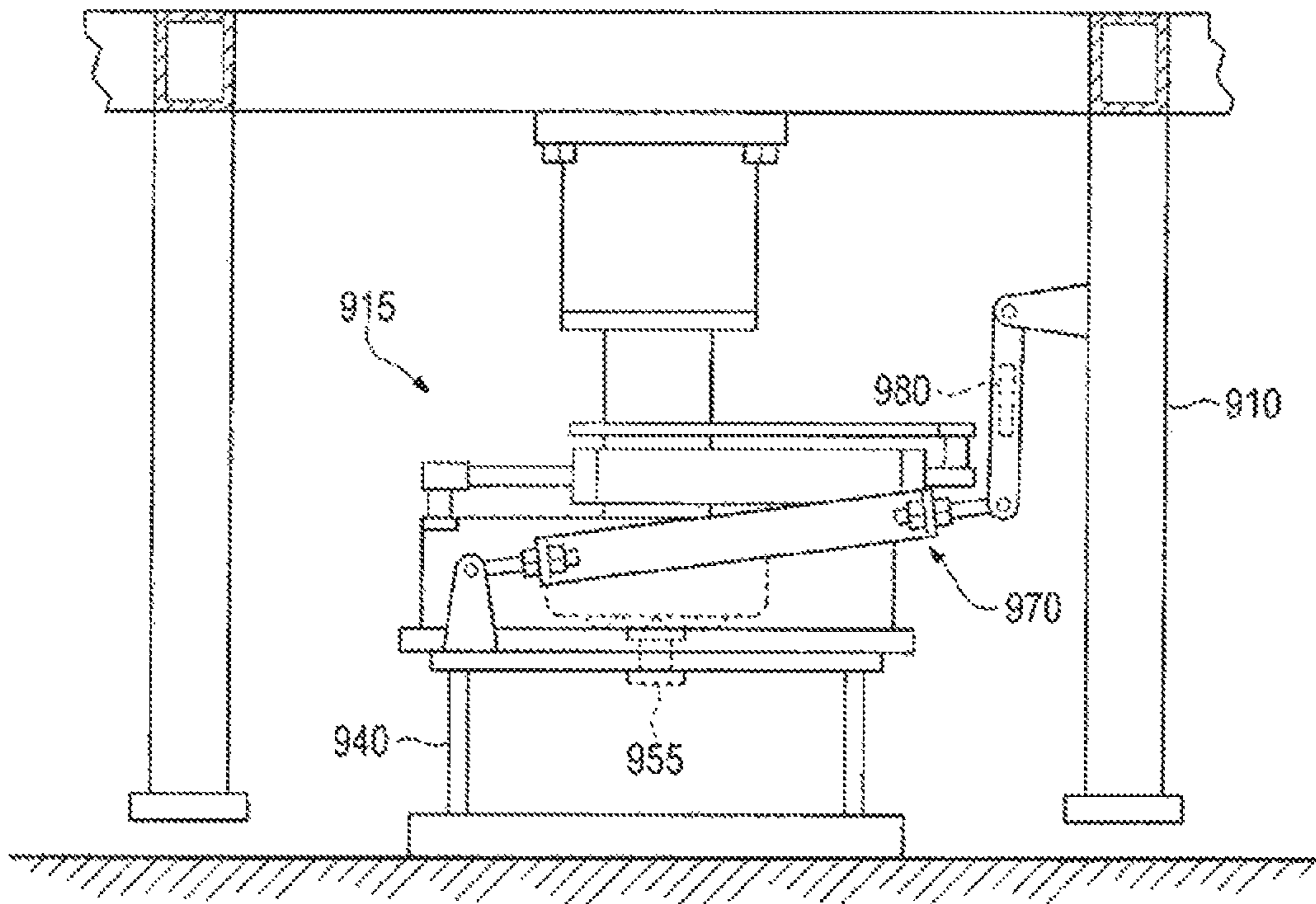


FIG. 9C

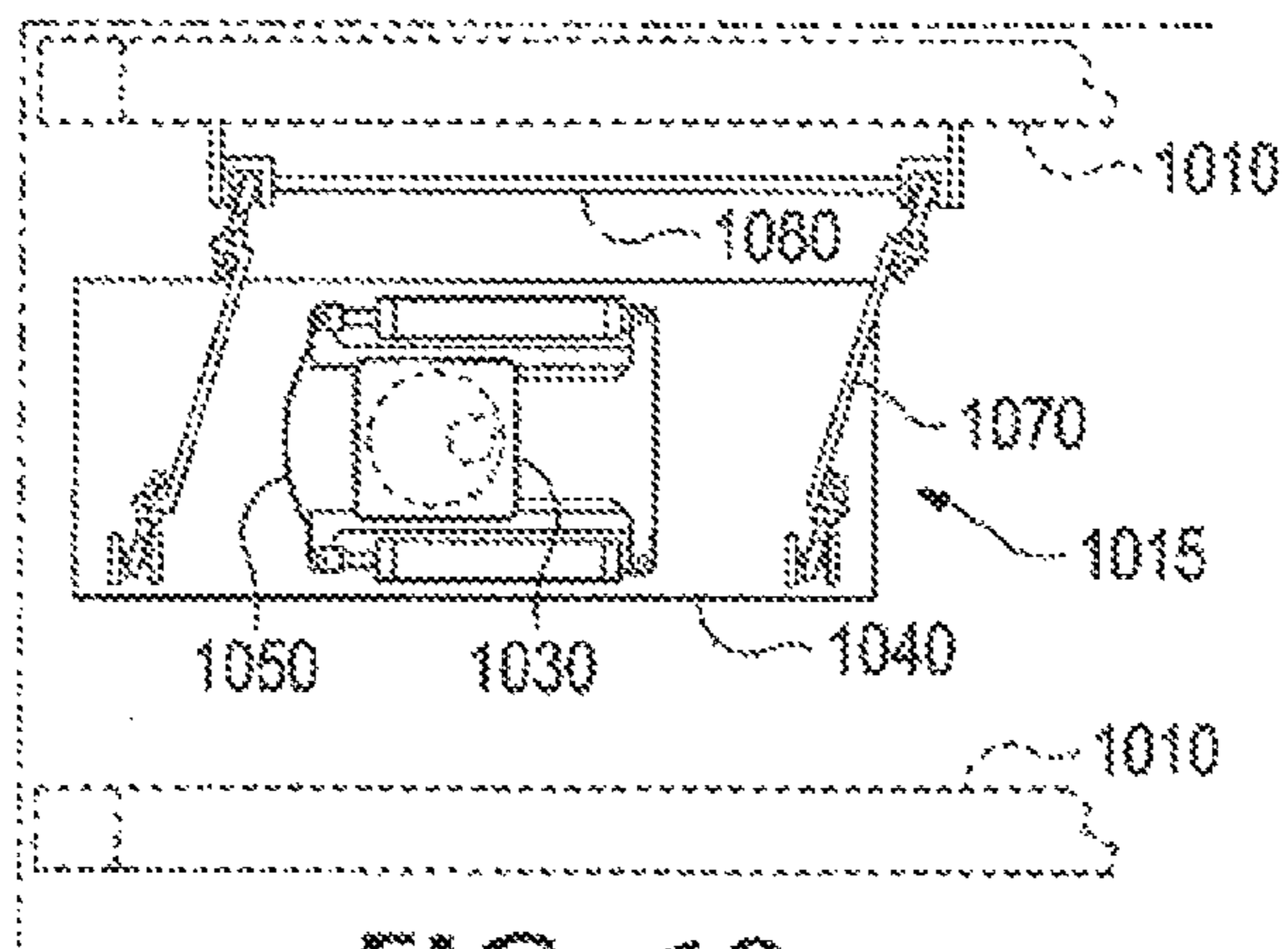


FIG. 10

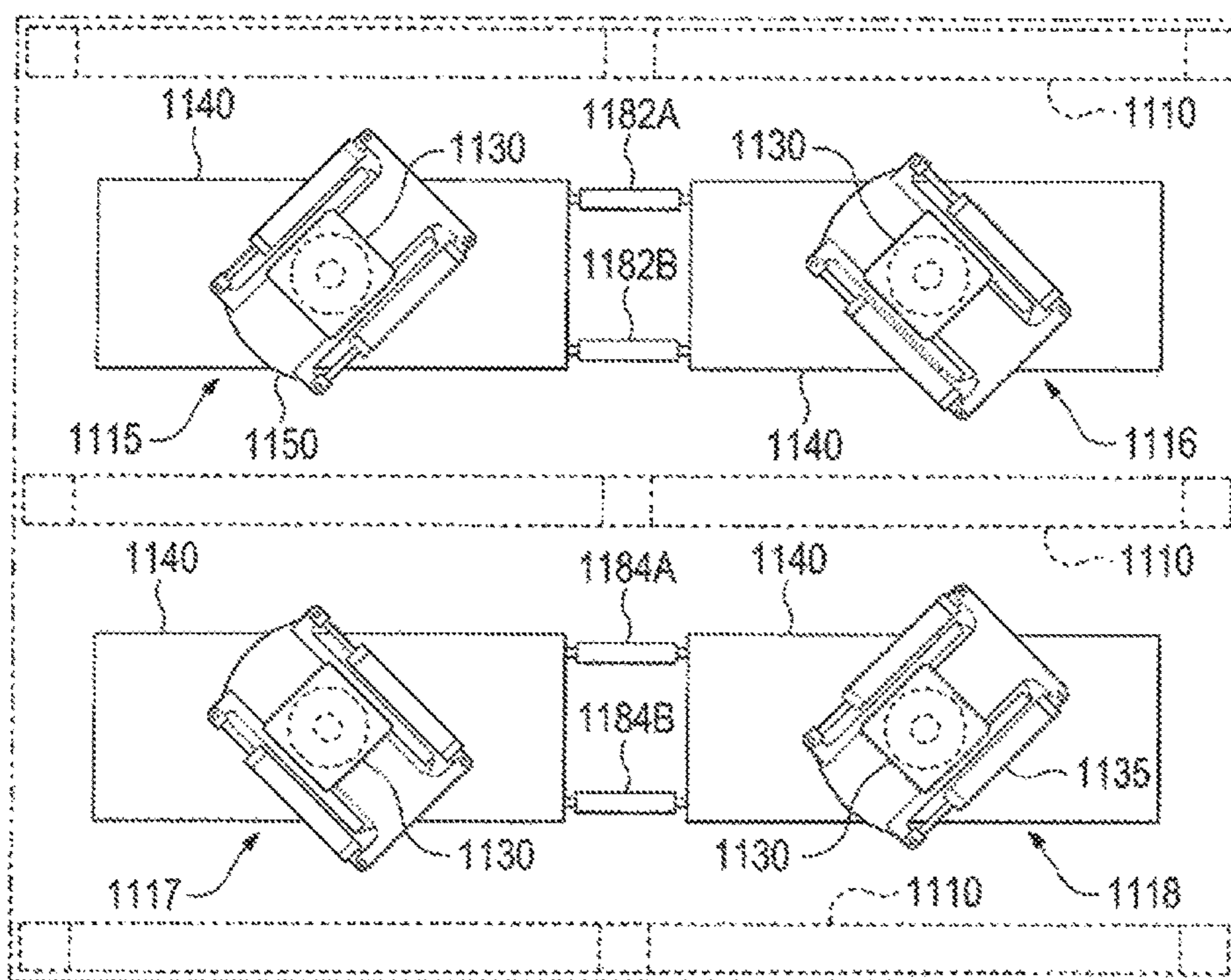
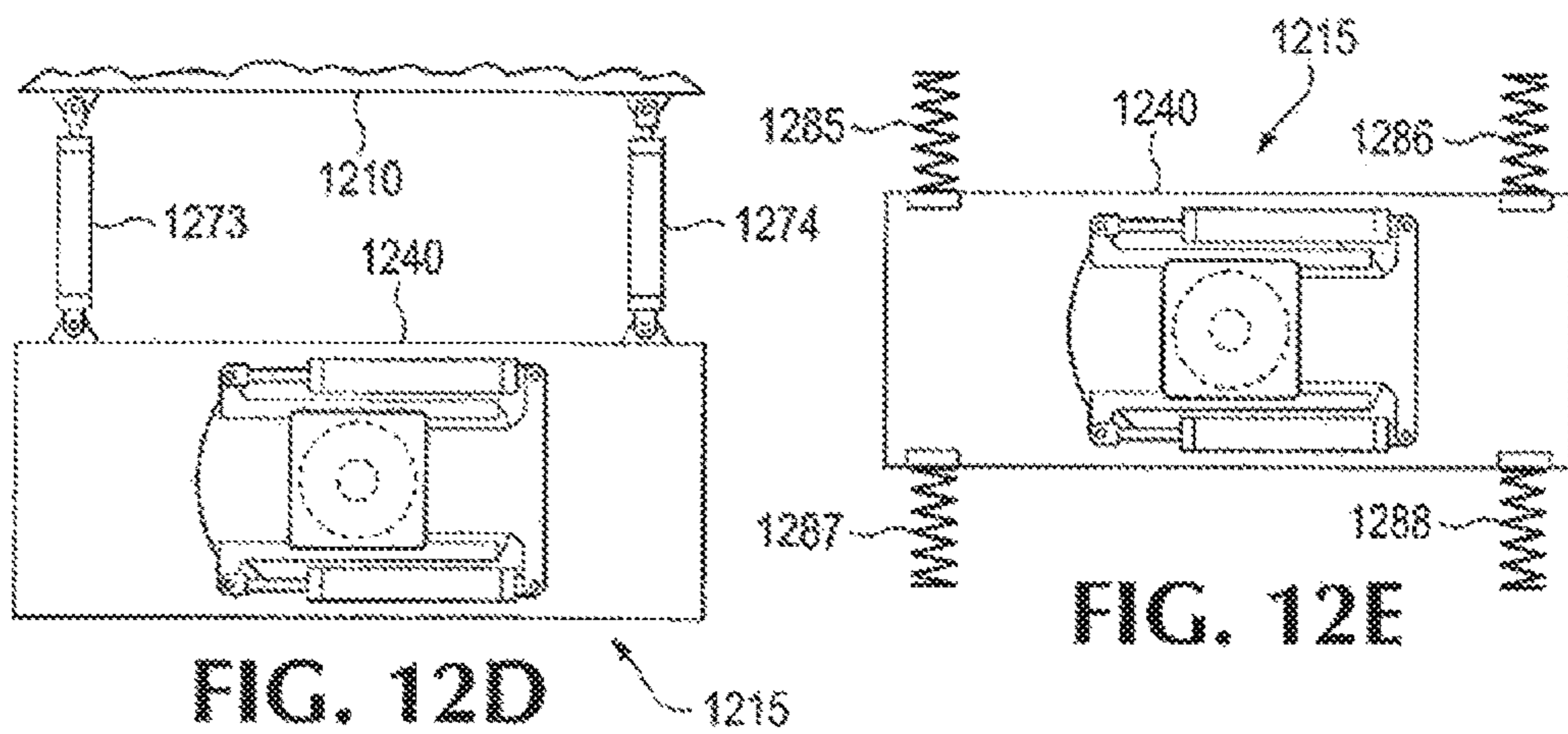
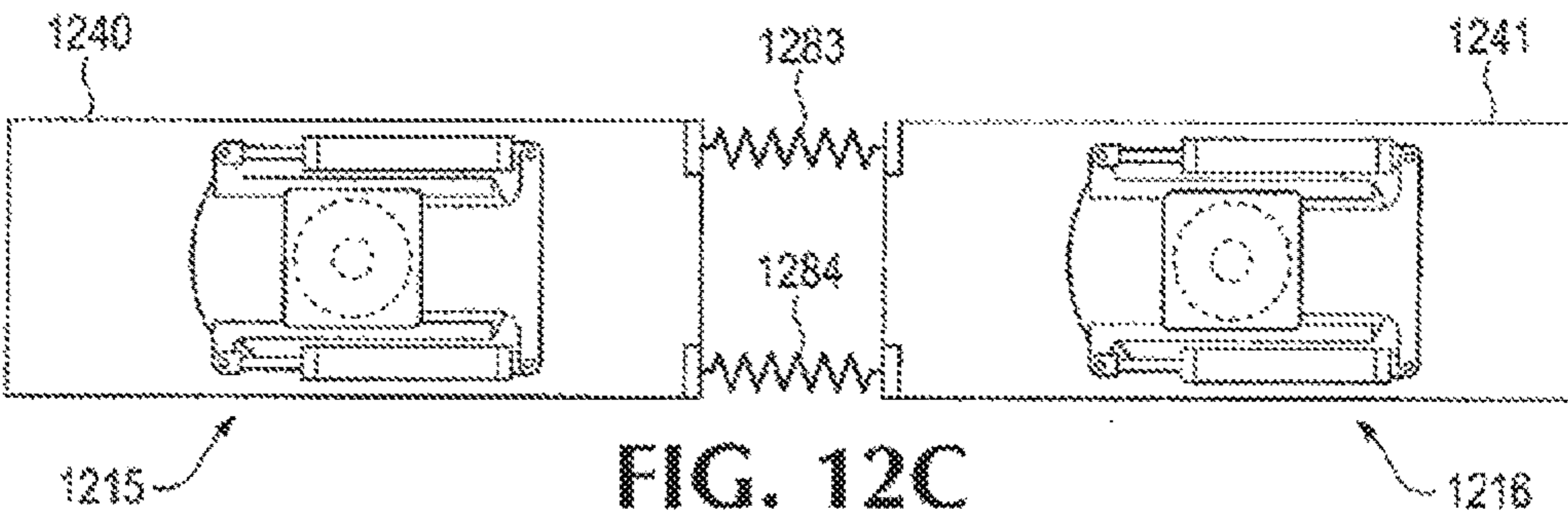
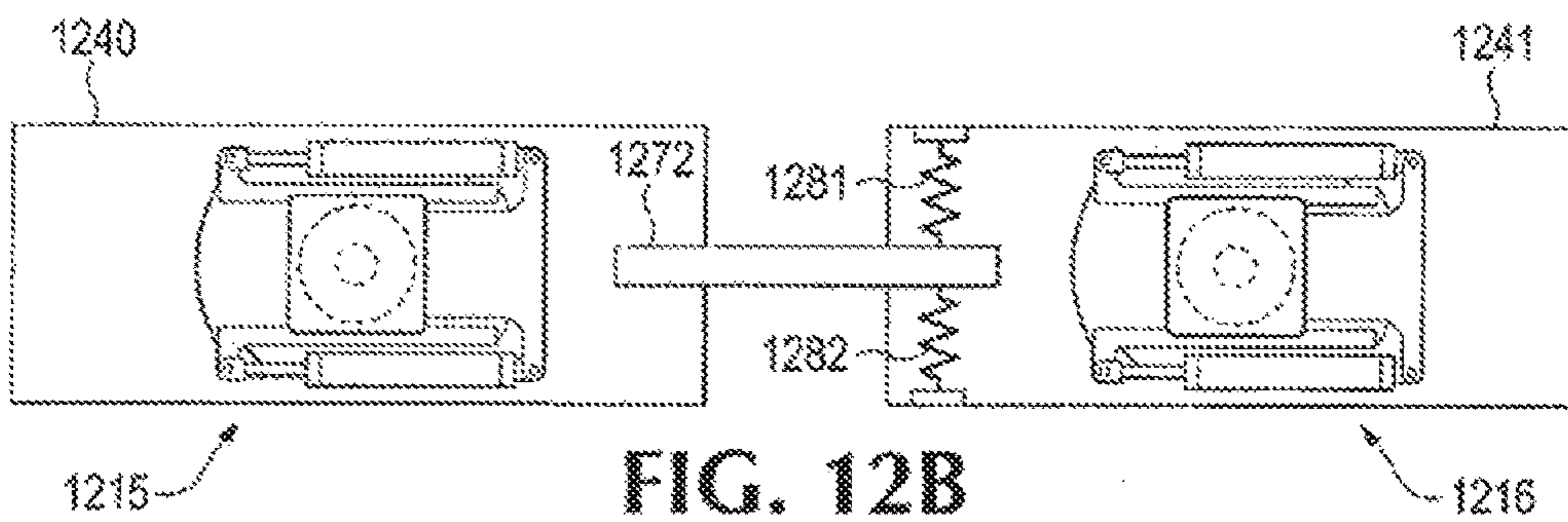
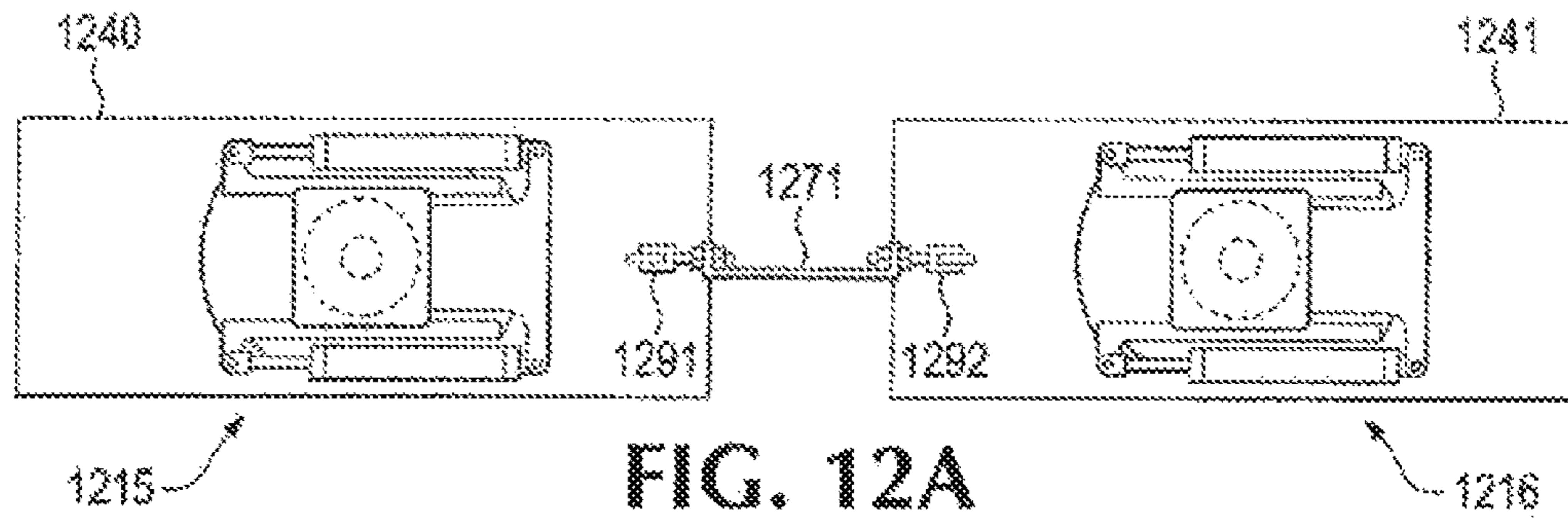


FIG. 11



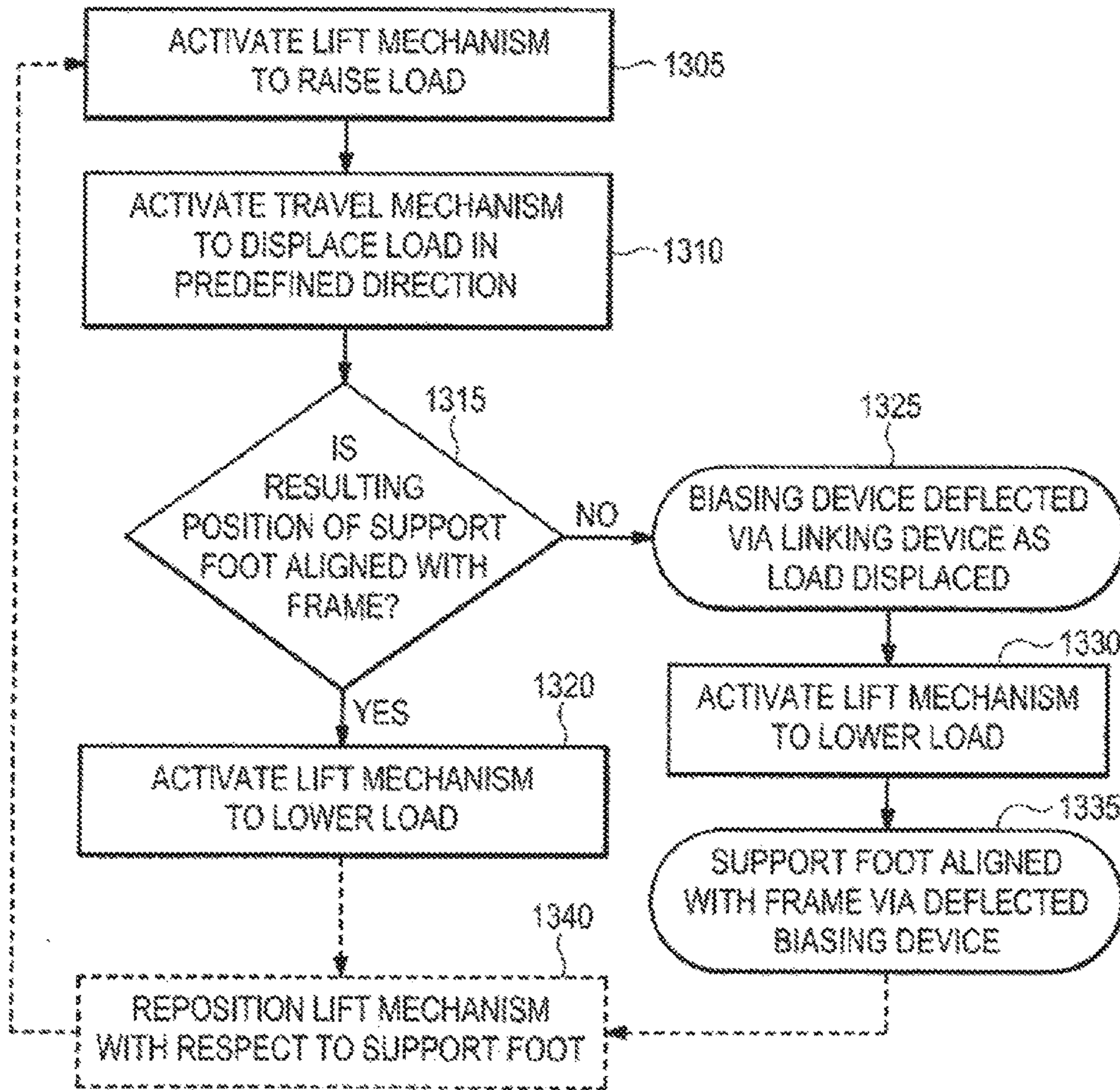


FIG. 13

ALIGNMENT RESTORATION DEVICE FOR LOAD TRANSPORTING APPARATUS

Matter enclosed in heavy brackets [] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue; a claim printed with strikethrough indicates that the claim was canceled, disclaimed, or held invalid by a prior post-patent action or proceeding.

RELATED APPLICATIONS

This application is a continuation of U.S. patent application Ser. No. 13/711,269, filed Dec. 11, 2012, now U.S. Pat. No. 8,561,733, issued Oct. 22, 2013, entitled ALIGNMENT RESTORATION DEVICE FOR LOAD TRANSPORTING APPARATUS, which claims priority to U.S. Provisional Application No. 61/576,657, filed Dec. 16, 2011, entitled METHOD AND APPARATUS FOR TRANSPORTING A LOAD, the contents of which are hereby incorporated by reference. This application is related to U.S. patent application Ser. No. 13/711,193, filed Dec. 11, 2012, now U.S. Pat. No. 8,573,334, issued Nov. 5, 2013, entitled ROTATION DEVICE FOR LOAD TRANSPORTING APPARATUS, the contents of which are hereby incorporated by reference. This application is also related to U.S. patent application Ser. No. 13,711,315, filed Dec. 11, 2012, now U.S. Pat. No. 8,490,724, issued Jul. 23, 2013, entitled CENTERING DEVICE FOR LOAD TRANSPORTING APPARATUS, the contents of which are hereby incorporated by reference.

FIELD OF THE INVENTION

This disclosure relates generally to apparatuses for transporting a load, and more particularly to apparatuses for moving heavy loads over small distances with the ability to fine tune the resultant position of the heavy load.

BACKGROUND

Moving extremely heavy loads has generally been a complicated task because the large forces involved in lifting and transporting the heavy loads. When possible, large loads are often transported by disassembling or breaking up the load into multiple smaller loads. However, this break-down and subsequent reassembly process can be very time consuming, especially when a heavy load is only to be moved a small distance, or needs to be repositioned.

For heavy loads that need periodic movement or adjustment, devices commonly referred to as “walking machines” or “walkers” were developed. These machines typically move the heavy loads over small distances in incremental stages. Walking machines are particularly useful for moving large structures, such as oil rigs, which often times need to be moved in order to properly position them over pre-drilled pipes in oil fields, or moved to a new location that is undergoing oil exploration.

Instead of using wheels driven by rotational forces to move heavy loads, walking machines typically use hydraulic lift cylinders to lift the load above a supporting surface, and then move or rotate the load relative to the supporting surface by transporting the load via rollers or tracks in the walking machines. U.S. Pat. No. 5,921,336 to Reed and U.S. Pat. No. 6,581,525 to Smith show two methods of using walking machines to move heavy loads, such as oil rig structures. The '525 patent shows elongated beams under

several rollers and lift cylinders, which allows the load from the lift cylinders and rollers to be spread over a large area. However, this disclosed system in the '525 patent does not allow for movement of heavy load in a direction perpendicular to the long axis of the support beams. This is, movement of the heavy load is restricted in the walking device disclosed in the '525 patent to only particular directions, which can make fine tuning of the position of the heavy load difficult.

SUMMARY

Embodiments of the present invention are directed to a load transporting apparatus that automatically aligns a support foot of the apparatus with a load-bearing frame connected to the load transporting apparatus during a recovery phase of an incremental walking movement. In particular, the load transporting apparatus includes a linking device attached to a support foot of the apparatus and a biasing device connected to the linking device that is deflected during non-linear load transporting movements, where the biasing device acts to automatically return the support foot to an aligned position relative to the load-bearing frame after a non-linear movement has been completed and the support foot is raised above a ground surface.

BRIEF DESCRIPTION OF THE DRAWINGS

FIGS. 1A and 1B are diagrams of walking apparatuses attached to various loads according to embodiments of the invention.

FIGS. 2A, 2B, 2C, 2D, 2E, and 2F are detail diagrams showing an example operational progression of walking apparatuses to move a load according to embodiments of the invention.

FIGS. 3A and 3B are diagrams illustrating example connection arrangements used to connect a walking apparatus to a load according to embodiments of the invention.

FIG. 4 is a schematic diagram illustrating movement of a load along a substantially linear path according to embodiments of the invention.

FIG. 5 is a schematic diagram illustrating movement of a load along a curved path according to embodiments of the invention.

FIG. 6 is a schematic diagram of a top view of a walking apparatus according to embodiments of the invention.

FIG. 7A is a side view of an example walking apparatus in a recovery position according to embodiments of the invention.

FIG. 7B is a side view of the example walking apparatus shown in FIG. 7A in a load-movement position according to embodiments of the invention.

FIGS. 8A, 8B, 8C, and 8D are side and top views of walking apparatuses that illustrate an example operation progression of a load transporting system according to embodiments of the invention.

FIG. 9A is a top view of a walking apparatus in a perpendicular orientation according to embodiments of the invention.

FIG. 9B is a side view of the walking apparatus shown in FIG. 9A in a load-movement position according to embodiments of the invention.

FIG. 9C is a side view of the walking apparatus shown in FIG. 9A in a recovery position according to embodiments of the invention.

FIG. 10 is a top view of a walking apparatus after a load-movement phase of a walking cycle completed in a parallel direction according to embodiments of the invention.

FIG. 11 is a top view of a load movement system according to embodiments of the invention.

FIGS. 12A, 12B, 12C, 12D, and 12E are diagrams of walking apparatuses with various alignment restoration devices according to embodiments of the invention.

FIG. 13 is a flow diagram illustrating method of operating a load transporting apparatus according to embodiments of the invention.

DETAILED DESCRIPTION

As described above, walkers, or walking machines, are vehicles that are used for transporting very heavy loads, such as entire oil well drilling rigs. Such loads may be as great as several thousand tons and may be required to be sequentially positioned very precisely over spaced-apart well bores, for example. Embodiments of the present concept are directed to load transporting apparatuses, such as walking machines, for moving heavy loads over small distances with the ability to fine tune the resultant position of the heavy load. For ease of understanding, the terms, "walkers," "walking machines," "walking devices," and "walking apparatuses" are used interchangeably below. Load transporting apparatuses or systems may include one or more walking machines. Additionally, a walking machine's subassembly of components that facilitate movement of the walking machine are referred herein as a "walking mechanism." Walking machines may incorporate one or more walking mechanisms, depending on the specific configuration of a walking machine.

For example, with reference FIGS. 1A and 1B, a load transporting system includes multiple walking machines that support a load being carried by the load transporting system. FIGS. 1A and 1B show examples of walking apparatuses attached to various loads according to embodiments of the invention. Referring to FIG. 1A, multiple walking apparatuses 115 are positioned under or adjacent to an oil rig 100. Typically, walking machines 115 are positioned at least near edge portions of a load 100 to balance the weight of the load over the various walking machines. However, specific situations may dictate that walking machines 115 are positioned in various other locations relative to the load 100.

Referring to FIG. 1B, multiple walking apparatuses 116 are positioned under or adjacent to a silo 101. Although an oil rig load 100 and a silo 101 are respectively illustrated in FIGS. 1A and 1B, walking machines may be used to move any type of relatively large load, such as bridge sections, ship sections, structures, etc. Additionally, although two walking machines are shown in FIGS. 1A and 1B, more or fewer walking machines may be used to move loads 100, 101.

FIGS. 2A-2F provide an overview of an example operation of walking apparatuses to move a load according to embodiments of the invention. Referring to FIG. 2A, walking apparatuses 215 are positioned on a base surface 205 below or adjacent to a load 200. Referring to FIG. 2B, the walking apparatuses 215 are attached to the load 200, and are positioned above a base surface 205. As described below, there are many possible connection variations that can be used to connect the walking apparatuses to a load 200. Referring to FIG. 2C, the walking apparatuses 215 are operated so that a foot portion of the walking apparatus contacts the base surface 205. The walking apparatuses 215

may be operated substantially simultaneously, or may be operated in intervals depending on the conditions of the base surface 205 and the load 200 that is to be moved.

Referring to FIG. 2D, the walking apparatuses 215 are operated to lift the load 200 above the base surface 205. The walking apparatuses 215 may again be operated substantially simultaneously to lift the load 200, or may be operated in intervals depending on the conditions associated with the desired move.

Referring to FIG. 2E, the walking apparatuses 215 are operated to move the load 200 to the right. Although FIG. 2E shows the load 200 being moved to the right, the walking apparatuses can be operated to move the load in a variety of directions depending on the desired final location of the load. Referring to FIG. 2F, the walking apparatuses 215 are operated to lower the load 200 to the base surface 205 and to raise the foot portions of the walking apparatuses above the base surface. That is, after the load 200 is positioned on the base surface 205, the walking apparatuses 215 are further operated so that they are raised above the base surface. Here, the connection between the walking apparatuses 215 and the load 200 support the walking apparatuses 215 when they are raised above the base surface 205. After the walking apparatuses 215 are raised above the base surface 205, they are further operated to be repositioned for another movement walking step, such as by moving the foot portions of the walking apparatuses to the right so that they are in a position as shown in FIG. 2B. That is, the base surface touching part of the walking apparatuses 215 (e.g., the support foot and related structures) is moved to the right while the walking apparatuses 215 are raised above the base surface 205. After the walking apparatuses 215 have been repositioned, they are operated to be lowered to the base surface 205 as shown in FIG. 2C. This completes a single walking cycle, and further walking cycles or steps can be performed by repeating the steps described above with respect to FIGS. 2D to 2F.

As mentioned above, walking apparatuses can be connected to loads in a variety of ways depending on the specific conditions surrounding the load. FIGS. 3A and 3B illustrate two such connection schemes. Although two connection schemes are illustrated in FIGS. 3A and 3B, embodiments of the invention are not limited to such connection schemes, as many different connection variations exist and are included in the scope of this concept.

Referring to FIG. 3A, a walking apparatus 315 includes a support foot 340 to interface with a base surface 305 and a lift mechanism 320 to raise and lower a load 300. In the embodiment shown in FIG. 3A, the lift mechanism 320 of the walking apparatus 315 is attached to a connection frame 318, which in turn is bolted to framework 310 supporting the load 300 with bolts 312 or other connection mechanisms. In some embodiments, the connection frame 318 may be part of the walking apparatus 315 and in some instances, may be permanently welded, bolted, or otherwise connected to the lift mechanism 320 of the walking apparatus. In other embodiments, the connection frame 318 may be separate from the walking apparatus 315, and may only be temporarily used with the walking apparatus in certain situations. In these embodiments, for example, multiple different connection frames 318 may be built or used with specific load conditions or specifications.

FIG. 3B shows different embodiments where the portions of a lift mechanism 320 of a walking apparatus 315 are directly connected to a support frame 310 structured to support a load 300 with bolts 312 or other connection mechanisms. The support frame 310 may be considered part of the load 300 in some instances where it is a permanent

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part of the load structure. For example, in instances where the load is a silo, such as shown in FIG. 1B, the metal frame of the silo may be considered the support frame 310 of the load 300, while also being part of the silo, and hence part of the load. In other cases, the support framework 310 may be an ancillary structure that is only used to stabilize and support the load 300 during movement of the load.

FIG. 4 is a schematic diagram illustrating movement of a load along a substantially linear path according to embodiments of the invention. Referring to FIG. 4, a load 400 is connected to multiple walking apparatuses 415, which are used to move the load from an initial position X_1 to a final position X_2 along a substantially linear path. Here, that path is a horizontal path moving from left to right. This type of basis linear movement can be accomplished by a variety of walking systems.

FIG. 5 is a schematic diagram illustrating movement of a load along a curved path according to embodiments of the invention. Referring to FIG. 5, a load 500 is connected to multiple walking apparatuses 515, which are used to move the load from an initial position X_3 to a final position X_4 along a non-linear path. Here, a reference center-point 502 of the load 500 at the initial position X_3 is moved to a reference center-point 592 of the load 500 at the final position X_4 . Unlike the linear movement shown in FIG. 4, this curved path of travel shown in FIG. 5 requires that the walking apparatuses be steered, which can be accomplished using embodiments of the inventive walking apparatuses described below.

FIG. 6 is a schematic diagram of a top view of a walking apparatus according to embodiments of the invention. Referring to FIG. 6, a load transporting apparatus 615 is configured to move a load (e.g., element 100 FIG. 1) over a base surface 605 in one or more incremental steps each including a load-movement phase and a recovery phase. The load transporting apparatus 615 includes a lift mechanism 620 structured to lift a load-bearing frame 610 supporting the load and a support foot 640 connected to the lift mechanism, the support foot structured to interface with the base surface 605. A roller assembly 630 is also coupled to the lift mechanism 620. A travel mechanism 660 is coupled to the roller assembly 630 and is structured to displace the roller assembly relative to the support foot 640. The load transporting apparatus also includes one or more linking devices 670 coupled to the support foot 640, and one or more biasing devices 680 coupled to the linking devices. The biasing devices 680 are structured to become activated during a load-movement phase when the roller assembly 630 is non-linearly displaced by the travel mechanism 660 relative to the support foot 640, and structured to return the support foot to an aligned position relative to the load-bearing frame 610 during a recovery phase. Here, the support foot 640 may be aligned with the load-bearing frame 610 when a longitudinal centerline of the support foot is parallel with a main beam of the load-bearing frame.

In these embodiments, the linking devices 670 are coupled to the biasing device 680 so that when the roller assembly 630 moves the load in a direction different than the orientation of the support foot 640, a deflection force is generated and/or stored as potential energy in the biasing device 680. This deflection force may be stored by deforming the biasing device 680 within the elastic region of a stress-strain curve associated with a material of the biasing device. For example, in embodiments where the biasing device 680 is a torsional bar, the deflection force transmitted to the biasing device during the non-linear displacement or movement may cause the torsional bar to twist.

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The contact between the support foot 640 and the base or ground surface 605 creates substantial frictional forces that prevent the support foot from rotating or moving during the non-linear displacement. During the recovery phase of the walking cycle, the support foot 640 is raised above the base surface 605, which eliminates the frictional forces between the foot and the base surface. Once the support foot 640 begins to lose contact with the base surface 605, the potential energy stored in the biasing device 680 is used to return the support foot to an aligned position relative to the load-bearing frame 610. The alignment of the load-bearing frame 610 is dictated by the movement of the roller assembly 630 by the travel mechanism 660. Hence, when the roller assembly 630 is non-linearly displaced (e.g., moved such as shown in FIG. 5), the orientation of the load-bearing frame 610 becomes skewed from the orientation of the support foot 640. In the above example, where the biasing device 680 is a torsional bar, the support foot 640 is returned to a positioned aligned relative to the load-bearing frame 610 when the support foot loses contact with the base surface 605 and the torsion bar is allowed to "untwist," thereby re-orienting the support foot. In other words, the torsion bar is activated when an angular displacement occurs between the support foot 640 and the load-bearing frame 610, where the activation of the torsion bar including a torquing force being applied to the torsion bar.

Although a torsion bar is discussed as the biasing device 680, may different types of biasing devices may be used in other embodiments, such as leaf springs, coil springs, chains, hydraulic cylinders, motors, or any other type of device that can be deflected and/or store potential energy to apply a realignment force to the support foot 640.

FIG. 6 is presented in a schematic style view as many possible variations in the appearance and mechanical structure of the load transporting apparatus 615 exist. FIGS. 7A and 7B provide a more detailed view of one embodiment of a load transporting apparatus. FIG. 7A is a side view of an example walking apparatus in a recovery position according to embodiments of the invention. FIG. 7B is a side view of the example walking apparatus shown in FIG. 7A in a load-movement position according to embodiments of the invention. Referring to FIGS. 7A and 7B, a load transporting or walking apparatus 715 includes a lift mechanism 720, a roller assembly 730, a roller track 750, and a support foot 740. The lift mechanism 720 may include a hydraulic jack suspended from a horizontal beam of the load-bearing frame 710. Additional details regarding the structure of the load transporting apparatus 715 can be found in co-pending application Ser. No. 13/711,193, entitled ROTATION DEVICE FOR LOAD TRANSPORTING APPARATUS, the contents of which are herein incorporated by reference in their entirety.

The roller track 750 of the walking apparatus 715 may be coupled to the support foot 740 with a connection mechanism that allows the support foot to rotate relative to the roller track. Various connection mechanisms may be used to facilitate this relative rotation, such as a rotation pin described below in FIG. 9 and in the above mentioned application Ser. No. 13/711,193. In addition, the lift mechanism 720 may be structured to allow the roller assembly 730 to rotate about a substantially vertical axis in the center of a cylinder rod of the lift mechanism. That is, the roller assembly 730 may also be free to rotate around the cylinder rod of the lift mechanism 720.

The walking apparatus 715 may also include a travel mechanism 760 that is connected to the roller track 750 and coupled to the roller assembly 730 such that when the travel

mechanism is activated, the roller assembly moves relative to the roller track. In the embodiment shown in FIGS. 7A and 7B, the travel mechanism 760 includes two travel cylinders mounted on the roller track 750 on opposite sides of the roller track. Here, the travel cylinders of the travel mechanism 760 may balance the load being moved by the roller assembly 730 over the roller track 750. In other embodiments, one travel cylinder, or three or more travel cylinders may be used to move the roller assembly 730 relative to the roller track 750. In other embodiments, the travel mechanism 760 may include different movement structures, such as pulleys, levers, winches, tracks, etc.

In the embodiments shown in FIGS. 7A and 7B, the roller assembly 730 may include a plurality of rollers or roller chain that rotate as well as roll on the roller track 750. That is, in some embodiments, the roller assembly 730 may include a WBOT series roller assembly from Hilman Rollers. Due to the configuration of the roller chain 730 of the roller assembly 730 and the tolerance between the roller assembly and the roller track 750 of the walking machine 715, the rollers of the roller chain will typically be engaged with the roller track during operation and use of the walking machine.

The roller assembly 730 may be secured to the lower end of the lift mechanism 720, with the roller assembly being captured within a U-shaped roller track 750. The roller assembly 730 may be configured to roll along the bottom inside surface of the roller track 750 as well as along the underside of the two upper flanges of the roller track. The one or more travel cylinders 760 may be coupled between the lift mechanism 720 and the roller track 750. Accordingly, as will be understood from the more detailed discussion below, these travel cylinders 760 permit for the translation of the roller track 750 relative to the lift mechanism 720 and vice versa. As discussed above, the roller track 750 may be secured to the elongate ground-engaging foot 740 (support foot) via a rotational pin (not shown in FIG. 7, but similar to element 955 of FIG. 9), which enables the roller track to be rotationally positioned relative to the foot for steering of the walking machine 715.

As shown in FIGS. 7A and 7B, a linking mechanism 770 is coupled to the support foot 740 and a biasing device 780 (shown more clearly as element 880) in FIG. 8A). In some embodiments, the linking mechanism 770 may include a first linking device attached at a first end of the support foot 740, where a second linking device connected to a second end of the support foot opposite of the first end of the first support foot (such as shown in FIGS. 6 and 8A). The biasing device 780 may be coupled between the first and second linking devices of the linking mechanism 770.

In the embodiments shown in FIGS. 7A and 7B, the linking mechanism 770 includes a first linking rod 772 connected to the support foot 740 with a first pivot joint 771. In some embodiments, the first pivot joint 771 may be a spherical rod end bearing configured to allow movement in three degrees of freedom. In other embodiments, the first pivot joint 771 may be another type of joint, such as a hinge joint, that restricts movement to one or two degrees of freedom.

The linking mechanism 770 may also include a second linking rod 774 connected to the first linking rod 772 with a second pivot joint 773. As with the first pivot joint 771, the second pivot joint 773 may be a spherical rod end bearing, or any other type of joint. The second linking rod 774 may further be connected to the load-bearing frame 710. In other embodiments, the one or more biasing devices 780 are also coupled to the load-bearing frame 710.

As shown in FIGS. 7A and 7B, the first and second pivot joints 771, 773 allow linking mechanism 770 to move vertically with the support foot 740 without deflecting or otherwise activating the biasing device 780.

As shown in co-pending application Ser. No. 13/711,315, entitled CENTERING DEVICE FOR LOAD TRANSPORTING APPARATUS, the contents of which is herein incorporated by reference in its entirety, a walking apparatus 715 may also include one or more guide devices positioned adjacent to the roller assembly 730, and one or more biasing devices coupled to the guide devices. Here, the biasing devices may be structured to become deflected during a load-movement phase when the movement of the roller assembly 730 deviates from a set direction of travel, and structured to return the support foot to a centered position relative to the support foot 740 during a recovery phase.

FIGS. 8A, 8B, 8C, and 8D are side and top views of walking apparatuses that illustrate an example operation progression of a load transporting system according to embodiments of the invention. Here, FIGS. 8A-8C may show a load-movement phase of a walking cycle, while FIG. 8D may show a recovery phase of a walking cycle, where the walking apparatus is in a spin steering mode.

Referring to FIG. 8A, a walking apparatus includes a support foot 840 positioned on a base surface 805 and connected to roller track 850. The roller track 850 is structured to allow a roller assembly 830 to move relative to the roller track when activated by a travel mechanism 860. A lift mechanism 820, such as hydraulic jack, is connected between the roller assembly 830 and load-bearing frame 810. A linking device 870 includes a first linking member 872 that is connected to the support foot, and a second linking member 874 that connects the first linking member to the load-bearing frame 810. A biasing device 880 is also connected to the linking device 870, and structured to become deflected or activated during a non-linear movement of the roller assembly 830 relative to the support foot 840. As shown in FIG. 8A, the walking apparatus 815 is in an initial position of a walking cycle in a spin steering mode. The roller tracks 850 of each walking apparatus 815 are oriented in a desired direction of travel. Here, in this first step of making a spin movement, the lift mechanisms 820 are activated to lift the load-bearing frame 810 (and load) above the base surface.

Referring to FIG. 8B, a step in a walking motion of the walking machine is illustrated. Specifically, as indicated by the arrows showing rotation of the load-bearing frame 810, the travel mechanism 860 is activated to displace the roller assembly 830 relative to the roller track 850 as shown. In this second step the walking system is moved in a circular or spin direction. Here, the travel cylinders of the travel mechanism 860 are actuated and the load-bearing frame 810 moves to a new angle. The support feet 840 are on the support surface and an angle of displacement occurs between the load-bearing frame 810 and the support feet. This non-linear movement or angular displacement causes an angular change in the biasing device 880. In embodiments where the biasing device 880 is a torsion bar, the resulting torque on the torsion bar causes the part of the linking device 870 to be in compression and causes another part of the linking device to be in tension.

Referring to FIG. 8C, the travel mechanism 860 has finished moving the roller assembly 830 and load-bearing frame 810. Additionally, the lift mechanism 820 has been activated to lower the load and load-bearing frame 810. Here, the load-bearing frame 810 has just contacted the ground surface. However, the support foot 840 is still

positioned on the ground surface as well. Hence, the biasing devices **880** are still in a deflected, activated, or biased state.

Referring to FIG. **8D**, the lift mechanism **820** is continued to be operated such that the support foot **840** loses contact with the ground surface. As soon as this connection between the support foot **840** and the ground surface disappears, the biasing device **880** causes the support foot to “snap” back into alignment with the load-bearing frame **810** as shown.

FIGS. **9A-9C** illustrate another embodiment of a walking apparatus. Here, FIG. **9A** is a top view of a walking apparatus in a perpendicular orientation according to embodiments of the invention. FIG. **9B** is a side view of the walking apparatus shown in FIG. **9A** in a load-movement position where the linking devices have been removed for clarity sake. FIG. **9C** is a side view of the walking apparatus shown in FIG. **9A** in a recovery position with the linking devices added back in for reference purposes.

Referring to FIGS. **9A-9C**, a walking apparatus **915** includes a lift mechanism **920** coupled to a load-bearing frame **910** that supports a load to be moved. The lift mechanism **920** is connected to a roller assembly **930** that is positioned on a roller track **950**. The roller assembly **930** is moved relative to the roller track **950** with one or more travel mechanisms **960**. The roller track **950** is coupled to a support foot **940** with a rotation pin **955**, such as a king pin or other connection means that allows rotation of the roller track relative to the support foot as described in the rotation device application (Ser. No. 13/711,193) cited above. A linking device **970** is coupled between the support foot **940** and the load-bearing frame **910**. A biasing device **980** is connected to the linking device **970**. As described above, the biasing device **980** becomes deflected or activated when the roller assembly **930** moves in a non-linear direction relative to the support foot **940**. For example, the roller track **950** is oriented perpendicular to the orientation of the support foot **940** in FIG. **9A**. As the roller assembly **930** moves in the direction of the orientation of the roller track **950**, the roller assembly and the load-bearing frame will also move substantially perpendicularly to the orientation of the support foot **940**.

Here, the movement of the roller assembly **930** in this orientation does not activate or deflect the biasing device **980** because the linking devices **970** include joints that allow for the free movement of the roller assembly. The linking devices **970** may be structured in this manner because the orientation of the support foot **940** relative to the load-bearing frame **910** does not change.

This can also be seen when the roller assembly is moved parallel to the orientation direction of the support foot, as shown in FIG. **10**. Referring to FIG. **10**, a walking apparatus **1015** has just completed a load-movement phase of a walking cycle where a roller track **1050** is oriented in the same direction as a support foot **1040**. Here, the roller assembly **1030** was moved to the right, along with the load-bearing frame **1010**, as shown. The joints of the linking device **1070**, however, allow the linking device to be angled from the linear movement without deflecting or otherwise activating the biasing device **1080**. During a recovery phase, the load-bearing frame **1010** is lowered and the support foot **1040** is raised above a base surface. The support foot **1040** can then be repositioned relative to the roller assembly **1030** by activation of the transport mechanism **960** (FIG. **9B**).

Some of the embodiments discussed above rely on the load-bearing frame as a reference point to realign the support feet during non-linear movements of the load. However, in other embodiments, other linking and biasing devices can

be utilized to maintain alignment of the support feet. Some of these techniques are discussed below with respect to FIGS. **11** and **12A-12E**.

FIG. **11** is a top view of a load movement system according to embodiments of the invention. Referring to FIG. **11**, multiple load transporting apparatuses **1115**, **1116**, **1117**, **1118** are used to move a load supported by a load-bearing frame **1110**. Each of these load transporting apparatuses **1115**, **1116**, **1117**, **1118** include a roller track **1150**, a roller assembly **1130** that moves relative to the roller track, and a support foot **1140**. Here, load transporting apparatuses that are in orientation-rows are connected with one or more biasing devices **1182**, **1184**. In particular, the support foot **1140** of a first load transporting apparatus **1115** is connected to the support foot of a second load transporting apparatus **1116** with two biasing devices **1182A** and **1182B**. These biasing devices **1182A**, **1182B** ensure that the first and second load transporting apparatuses **1115**, **1116** are maintained in alignment with one another and the load-bearing frame **1110**.

Here, the linking devices include a first linking device **1182A** coupled between a first side of a first end of the first support foot **1140** and a first side of a first end of the second support foot **1140**, and a second linking device **1182B** coupled between a second side of the first end of the first support foot and a second side of the first end of the second support foot. The placement of the first and second linking devices **1182A**, **1182B** may ensure that the support feet **1140** are aligned together during a non-linear movement.

Similarly, the support foot **1140** of a third load transporting apparatus **1117** is connected to the support foot of a fourth load transporting apparatus **1118** with two biasing devices **1184A** and **1184B**. These biasing devices **1184A**, **1184B** ensure that the third and fourth load transporting apparatuses **1117**, **1118** are maintained in alignment with one another and the load-bearing frame **1110**.

Although FIG. **11** illustrates one example embodiment of biasing device connections that can maintain alignment of a support foot relative to a load-bearing frame, many different configuration variations exist. FIGS. **12A**, **12B**, **12C**, **12D**, and **12E** are diagrams of walking apparatuses with various alignment restoration devices that illustrate some of these variations according to embodiments of the invention.

Referring to FIG. **12A**, a linking device **1271** is connected between a first support foot **1240** of a first load transporting apparatus **1215** and a second support foot **1241** of a second load transporting apparatus **1216**. The linking device **1271** may be attached to the first support foot **1240** with a first joint **1291**, and may be attached to the second support foot **1241** with a second joint **1292**. In some embodiments, the first and second joints **1291**, **1292** may be ball joints that allow rotational movement. The linking device **1271** may be rigid rod, or may include a section of chain.

Referring to FIG. **12B**, a linking device **1272** is connected between a first support foot **1240** of a first load transporting apparatus **1215** and a second support foot **1241** of a second load transporting apparatus **1216**. The linking device **1272** may be rigidly attached to the first support foot **1240**, but may be attached to the second support foot **1241** with a first biasing device **1281** and a second biasing device **1282**. The first and second biasing devices **1281**, **1282** may be placed on opposite sides of the linking device **1272** to provide a balanced system to return the support feet **1240**, **1241** to uniform alignment after a non-linear movement.

Referring to FIG. **12C**, a first biasing device **1283** and a second biasing device **1284** are connected between a first support foot **1240** of a first load transporting apparatus **1215**

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and a second support foot 1241 of a second load transporting apparatus 1216. This embodiment may be similar to the shown in FIG. 11, except that the first and second biasing devices 1283, 1284 are specified as spring devices.

Referring to FIG. 12D, the support foot 1240 of a load transporting apparatus 1215 is connected to a load-bearing frame 1210 via a first linking cylinder 1273 and a second linking cylinder 1274. The first and second linking cylinders 1273, 1274 may be hydraulic cylinders that are activated during a recovery phase of a walking cycle to return the support foot 1240 to alignment with the load-bearing frame 1210. Alternatively, the first and second linking cylinders 1273, 1274 may be spring cylinders that automatically return the support foot 1240 to alignment with the load-bearing frame 1210 during a recovery phase of a walking cycle without additional operator input.

Referring to FIG. 12E, a support foot 1240 of a load transporting apparatus 1215 is connected at each corner to a biasing device 1285, 1286, 1287, 1288. These biasing devices 1285, 1286, 1287, 1288 may ensure that the support foot 1240 is maintained in alignment with a load-bearing frame during the recovery phase of a walking cycle by releasing potential energy stored during compression and/or elongation during non-linear movements.

FIG. 13 is a flow diagram illustrating method of operating a load transporting apparatus according to embodiments of the invention. In particular, the flow diagram of FIG. 13 illustrates a method of aligning a support foot of a load transporting device relative to a load-bearing frame during a load-transporting movement. The load transporting device includes a roller assembly coupled to a lift mechanism, a travel mechanism structured to displace the roller assembly relative to the support foot, one or more linking devices coupled to the support foot, and one or more biasing devices coupled to the linking devices.

Referring to FIG. 13, a flow begins at process 1305 where the lift mechanism is activated to lower the support foot to a ground surface and raising a load supported by the load-bearing frame. In process 1310, the travel mechanism is activated to displace the roller assembly connected to the lift mechanism relative to the support foot and ground surface, thereby moving a position of the load. Depending on the movement of the travel mechanism relative to the support foot, the position of the support foot may be aligned with the load-bearing frame or may not be aligned with the load-bearing frame. As discussed above, when the load is moved in a direction perpendicular to the orientation of the support foot, or moved parallel to the orientation of the support foot, the support foot typically remains aligned with the load-bearing-frame. If the load is moved in a different direction relative to the support foot, such as when the load is being steered in a non-linear path, the support foot can become misaligned with the load-bearing frame. In process 1315, it is observed whether the resulting position of the support foot is aligned with the load-bearing frame.

When the support foot remains aligned with the load-bearing frame, the flow proceeds to process 1320 where the lift mechanism is activated to lower the load and raise the support foot. However, when the support foot is not aligned with load-bearing frame, the biasing device is deflected via the linking device as the load is displaced as shown in step 1325. That is, the biasing devices are deflected when movement of the roller assembly results in an angular displacement between a centerline of the support foot and an orientation of the load-bearing frame. In process 1330, the lift mechanism is activated to lower the load and raise the support foot from the ground surface. As the support foot

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loses contact with the ground surface, the deflected biasing device acts on the support foot to align the support foot with the load-bearing frame, as shown in step 1335. That is, the centerline of the support foot is automatically aligned relative to the orientation of the load-bearing frame. After step 1335 or process 1320, the flow may include optional process 1340 where the lift mechanism is repositioned with respect to the support foot. If further walking steps are needed to move the load to a final position, the flow may return to process 1305 to initiate another walking cycle.

As described above, some embodiments of this invention are directed to a load transporting apparatus configured to move a load over a ground surface in one or more incremental steps each including a load-movement phase and a recovery phase. To move the load, the load transporting apparatus is coupled to a load-bearing frame configured to support the load. The load transporting apparatus includes a first support foot structured to interface with the ground surface, the first support foot having a length, width, and longitudinal centerline bisecting the width of the first support foot. The load transporting apparatus also includes a second support foot structured to interface with the ground surface, the second support foot also having a length, width, and longitudinal centerline bisecting the width of the second support foot.

First and second roller tracks are respectively coupled to the first support foot and second support foot via a first king pin connector and a second king pin connector. Additionally, first and second roller assemblies are respectively positioned on the first and second roller tracks. Each roller assembly includes a roller frame and one or more rollers set in the roller frame. First and second lift mechanisms are respectively coupled to the first and second roller assemblies. Each of the first and second lift mechanisms includes a lift cylinder connected to the load-bearing frame, and a cylinder rod, where each of the first and second lift mechanisms are structured to lift the load-bearing frame at the start of the load-movement phase.

The load transporting apparatus also includes first and second travel mechanisms respectively coupled to the first and second roller assemblies. Each of the travel mechanisms are structured to move the respective roller assembly relative to the respective support foot during the load-movement phase. A first linking device coupled to the first support foot, and a second linking device coupled to the second foot. A first biasing device is connected to the first linking device, where the first biasing device is structured to become activated during a load-movement phase when the first roller assembly is non-linearly displaced by the first travel mechanism relative to the first support foot, and structured to return the first support foot to an aligned position relative to the load-bearing frame during a recovery phase. A second biasing device is connected to the second linking device, where the second biasing device is structured to become activated during a load-movement phase when the second roller assembly is non-linearly displaced by the second travel mechanism relative to the second support foot, and structured to return the second support foot to an aligned position relative to the load-bearing frame during a recovery phase.

In some embodiments, the first linking device is coupled between the first support foot and the second support foot. In these embodiments, the second linking device is also coupled between the first support foot and the second support foot, as shown in FIG. 11, for example. In other

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embodiments, the first and second biasing devices are respectively coupled to the load-bearing frame, such as in FIG. 7A, for example.

Some embodiments of the invention have been described above, and in addition, some specific details are shown for purposes of illustrating the inventive principles. However, numerous other arrangements may be devised in accordance with the inventive principles of this patent disclosure. Further, well known processes have not been described in detail in order not to obscure the invention. Thus, while the invention is described in conjunction with the specific embodiments illustrated in the drawings, it is not limited to these embodiments or drawings. Rather, the invention is intended to cover alternatives, modifications, and equivalents that come within the scope and spirit of the inventive principles set out herein.

The invention claimed is:

1. A load transporting apparatus comprising:
 - a support foot comprising a longitudinal centerline;
 - a roller assembly operably coupled to the support foot and configured to move relative to the support foot;
 - a lift mechanism operably coupled to the roller assembly and configured to lift [the] a load [transporting apparatus] *bearing frame* from a transport surface;
 - a travel mechanism operably coupled to the roller assembly and configured to non-linearly displace the support foot such that the longitudinal centerline of the support foot moves relative to a longitudinal centerline of the load [transporting apparatus] *bearing frame*; and
 - one or more biasing devices configured to elastically and materially deform in response to the non-linear displacement of the support foot, wherein an elastic deformation of the one or more biasing devices provides a biasing force that operates to realign the support foot with the load [transporting apparatus] *bearing frame* such that the longitudinal centerline of the support foot becomes parallel with the longitudinal centerline of the load [transporting apparatus] *bearing frame* after the load [transporting apparatus] *bearing frame* is lowered to the transport surface.
2. The apparatus of claim 1, further comprising one or more linking devices operably coupling the support foot to the one or more biasing devices.
3. The apparatus of claim 2, wherein the biasing devices are further operably coupled to a load-bearing frame.
4. The apparatus of claim 2, wherein the one or more linking devices includes a first linking device attached at a first end of the support foot, and a second linking device connected to a second end of the support foot opposite of the first end of the support foot.
5. The apparatus of claim 4, wherein the one or more biasing devices include a torsion bar coupled between the first linking device and the second linking device.
6. The apparatus of claim 5, wherein the torsion bar is configured to undergo a torqueing force in response to the non-linearly displacement of the longitudinal centerline of the support foot relative to the longitudinal centerline of the load [transporting apparatus] *bearing frame*.
7. [An apparatus having a] A load transporting [device, a roller assembly, and a support foot, the] apparatus comprising:
 - a roller assembly;
 - a support foot;
 - means for raising [the load transporting device] a load *bearing frame* from a transport surface;
 - means for moving the roller assembly relative to the support foot [of the load transporting device], wherein

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the roller assembly is operably coupled to the means for raising, and wherein the load [transport device] *bearing frame* moves in response to moving the roller assembly;

means for biasing; and

means for deflecting the means for biasing in response to moving the roller assembly, wherein the movement of the roller assembly results in a non-linear displacement between a centerline of the support foot and a centerline of the load [transporting device] *bearing frame*, wherein the means for biasing is configured to elastically and materially deform in response to being deflected, and wherein an elastic deformation of the means for biasing devices provides a biasing force that operates to realign the centerline of the support foot relative to the centerline of the load [transporting device] *bearing frame* following the movement of the roller assembly.

8. The apparatus of claim 7, wherein the means for raising comprises means for raising the support foot from the transport surface, and wherein in response to raising the support foot, the means for biasing is configured to move the support foot relative to the load [transporting device] *bearing frame* such that the centerline of the support foot is realigned relative to the centerline of the load [transporting device] *bearing frame*.

9. The apparatus of claim 7, wherein the means for biasing is operably coupled to [a] the load-bearing frame.

10. The apparatus of claim 7, further comprising means for preventing activation of the means for biasing when the support foot is moved such that the centerline of the support foot is linearly displaced by the means for deflecting relative to the centerline of the load [transporting device] *bearing frame*.

11. The apparatus of claim 7, wherein the means for deflecting comprises means for linking the support foot to the means for biasing, and wherein the means for linking comprises:

- first means for linking operably coupled at a first end of the support foot; and
- second means for linking operably coupled to a second end of the support foot.

12. The apparatus of claim 11, wherein the means for biasing comprises a torsion bar coupled between the first means for linking and the second means for linking.

13. The apparatus of claim 12, wherein the torsion bar is configured to undergo a torqueing force in response to the non-linearly displacement of the longitudinal centerline of the support foot relative to the longitudinal centerline of the load [transporting apparatus] *bearing frame*.

14. A method comprising:

- raising, with a lift mechanism, a load [transporting device] *bearing frame* from a transport surface;
- moving a roller assembly relative to a support foot [of the load transporting device], wherein the roller assembly is operably coupled to the lift mechanism, and wherein the load [transport device] *bearing frame* moves in response to moving the roller assembly;
- angularly displacing a centerline of the support foot and a centerline of the load [transporting device] *bearing frame*;
- deflecting one or more biasing devices in response to moving the roller assembly, wherein the one or more biasing devices are configured to elastically and materially deform in response to being deflected; and
- realigning the centerline of the support foot relative to the centerline of the load [transporting device] *bearing*

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frame in response to a biasing force provided by an elastic deformation of the one or more deflected biasing devices.

15. The method of claim 14, further comprising lowering the support foot to the transport surface prior to raising the load [transporting device] *bearing frame*.

16. The method of claim 15, further comprising raising the support foot from the transport surface after moving the roller assembly relative to the support foot, wherein the centerline of the support foot is realigned relative to the centerline of the load [transporting device] *bearing frame* in response to raising the support foot.

17. The method of claim 14, wherein the one or more biasing devices comprise a torsion bar, and wherein deflecting the one or more biasing devices comprises applying a torque force to the torsion bar.

18. The method of claim 17, wherein realigning the centerline of the support foot comprises applying the torque force to the support foot via one or more linking devices operably coupling the support foot to the one or more biasing devices.

19. The method of claim 18, wherein the torsion bar is operably coupled between a first linking device and a second linking device of the one or more linking devices.

20. The method of claim 19, wherein the first linking device is operably coupled at a first end of the support foot, and wherein the second linking device is operably coupled to a second end of the support foot opposite of the first end.

21. *A load transporting apparatus comprising:*

a support foot;

a roller assembly operably coupled to the support foot;

a lift mechanism operably coupled to the roller assembly and configured to lift a load bearing frame from a transport surface;

a travel mechanism configured to displace the support foot relative to the lift mechanism, wherein an activation of the travel mechanism operates to non-linearly displace the support foot relative to an orientation of the load bearing frame; and

one or more biasing devices configured to elastically and materially deform in response to the non-linear displacement of the support foot, wherein an elastic defor-

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mation of the one or more biasing devices provides a biasing force that operates to realign the support foot with the orientation of the load bearing frame.

22. *The load transporting apparatus of claim 21, wherein the one or more biasing devices comprise a torsion bar substantially aligned with the orientation of the load bearing frame.*

23. *The load transporting apparatus of claim 21, further comprising one or more linking devices operably coupled to both the support foot and the one or more biasing devices.*

24. *The load transporting apparatus of claim 23, wherein the one or more linking devices comprise a first linking device operably coupled to a first end of the support foot, and a second linking device operably coupled to a second end of the support foot opposite the first end of the support foot.*

25. *The load transporting apparatus of claim 24, wherein the one or more biasing devices comprise a torsion bar operably coupled to both the first linking device and the second linking device.*

26. *The load transporting apparatus of claim 25, wherein the torsion bar is configured to undergo a torqueing force in response to the non-linearly displacement of the support foot relative to the orientation of the load bearing frame.*

27. *The load transporting apparatus of claim 21, wherein the one or more biasing devices are configured to elastically and materially deform when the support foot is in a lowered position on the transport surface, and wherein the biasing force operates to realign a centerline of the support foot with the orientation of the load bearing frame when the support foot is raised from the transport surface.*

28. *The load transporting apparatus of claim 27, wherein the one or more biasing devices are configured to elastically and materially deform with the load bearing frame raised above the transport surface, and wherein the biasing force operates to realign the centerline of the support foot with the orientation of the load bearing frame after the load bearing frame is lowered to the transport surface.*

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UNITED STATES PATENT AND TRADEMARK OFFICE
CERTIFICATE OF CORRECTION

PATENT NO. : RE46,723 E
APPLICATION NO. : 15/141587
DATED : February 20, 2018
INVENTOR(S) : Smith et al.

Page 1 of 1

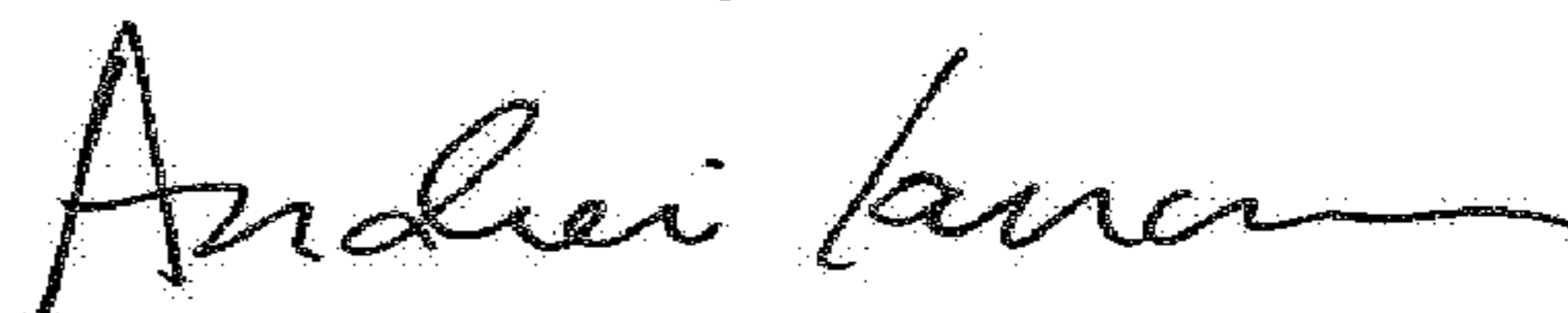
It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

In the Specification

Please insert Line 11 (approx.) of Column 1 as follows:

--Notice: More than one reissue application has been filed for the reissue of U.S. Patent No. 9,004,203. The reissue applications are U.S. Reissue Patent Application Serial No. 15/867,382, filed on January 10, 2018, which is a continuation reissue application of U.S. Reissue Patent Application Serial No. 15/141,587 (the present application), filed on April 28, 2016, now U.S. Reissue Patent No. RE46,723 E, issued February 20, 2018.--

Signed and Sealed this
Twelfth Day of June, 2018



Andrei Iancu
Director of the United States Patent and Trademark Office