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**Weissert**

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- (54) **COMPLIANT FOIL FLUID FILM RADIAL BEARING**
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- (73) Assignee: **Capstone Turbine Corporation**, Chatsworth, CA (US)
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4,153,315 A	5/1979	Silver	
4,170,389 A	10/1979	Eshel	
4,178,046 A	12/1979	Silver	
4,195,395 A	4/1980	Silver	..... 29/149.5 S
4,208,076 A	6/1980	Gray	
4,213,657 A	7/1980	Gray	
4,223,958 A	9/1980	Gray	
4,229,054 A	10/1980	Miller	
4,247,155 A	1/1981	Fortmann	
4,262,975 A	4/1981	Heshmat	
4,295,689 A	10/1981	Licht	
4,300,806 A	11/1981	Heshmat	
4,315,660 A	2/1982	Glienicke	
4,371,216 A	2/1983	Suzuki	..... 384/118
4,415,280 A	11/1983	Agrawal	..... 384/103
4,415,281 A	11/1983	Agrawal	..... 384/106
4,435,839 A	3/1984	Gu	..... 384/103

**Related U.S. Patent Documents**

Reissue of:

- (64) Patent No.: **5,915,841**
- Issued: **Jun. 29, 1999**
- Appl. No.: **09/002,690**
- Filed: **Jan. 5, 1998**

- (51) **Int. Cl.**  
**F16C 17/03** (2006.01)
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  - (58) **Field of Classification Search** ..... 384/104,  
384/106, 103, 105
- See application file for complete search history.

(56) **References Cited**

**U.S. PATENT DOCUMENTS**

3,366,427 A	1/1968	Silver
3,382,014 A	5/1968	Marley
3,615,121 A	10/1971	Barnett
3,635,534 A	1/1972	Barnett
3,747,997 A	7/1973	Winn
3,795,427 A	3/1974	Licht
3,809,443 A	5/1974	Cherubim
3,957,317 A	5/1976	Silver
4,118,079 A	10/1978	Newman
4,133,585 A	1/1979	Licht

(Continued)

**OTHER PUBLICATIONS**

“Foil Air/Gas Bearing Technology—An Overview,” Giri Agrawal, *The American Society of Mechanical Engineers*, Jun. 1997, pp. 1–11.

“Foil Gas Bearings for Turbomachinery,” Giri Agrawal, *The Engineering Society for Advancing Mobility Land Sea Air and Space, SAE Technical Paper Series*, Jul. 1990.

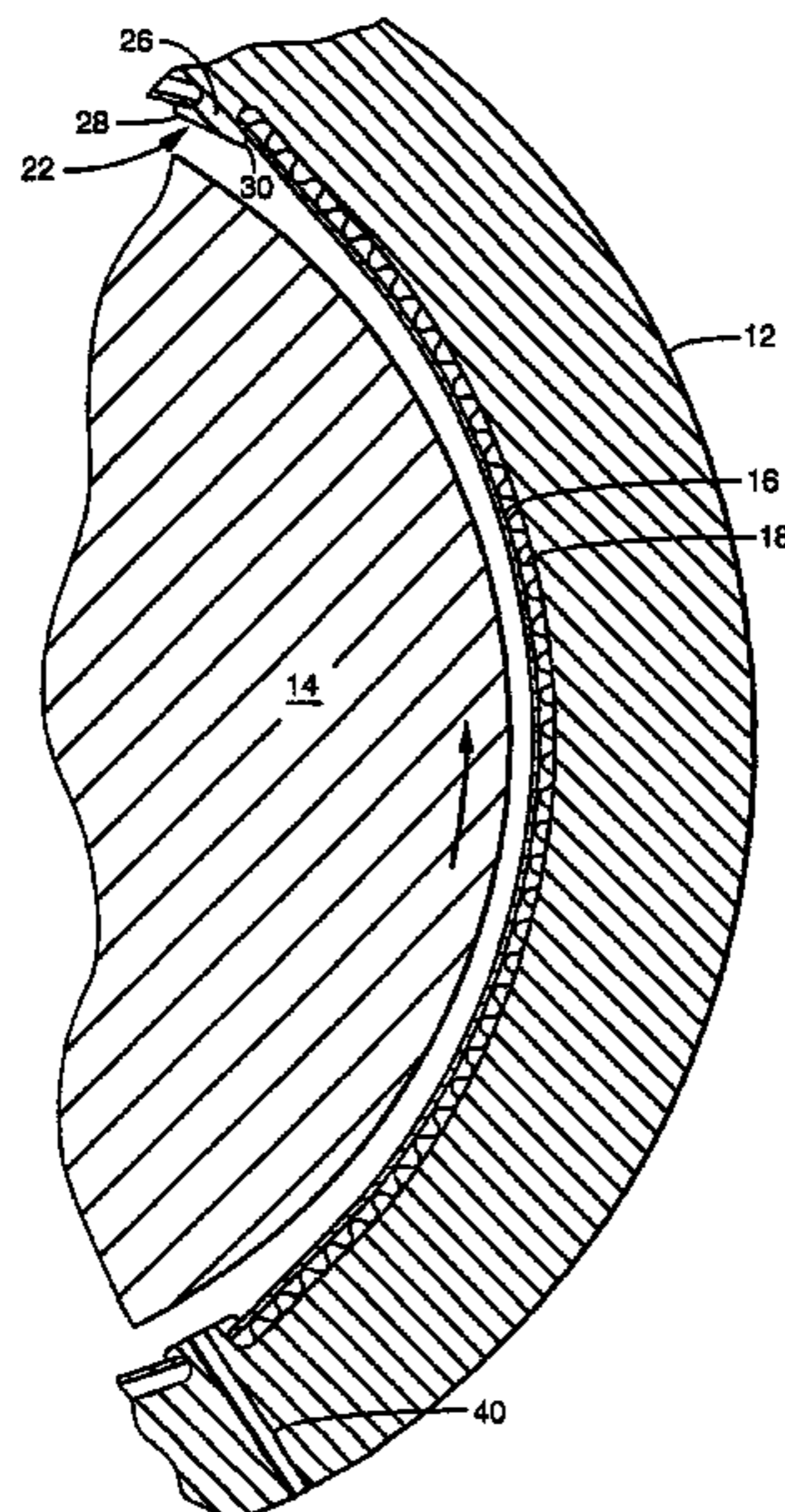
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Lucian Wayne Beavers

(57) **ABSTRACT**

A multi-segment radial bearing including a bushing with an interior bore having a plurality of anti-rotation retainers which are equally spaced and extend the axial length of the interior bore. The generally T-shaped retainers divide the interior bore of the bushing into a like plurality of lobes, with each lobe having a compliant foil and a foil underspring disposed between adjacent generally T-shaped retainers.

**56 Claims, 3 Drawing Sheets**



# US RE39,190 E

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## U.S. PATENT DOCUMENTS

4,445,792 A	5/1984	Trippett	384/103	5,248,205 A	9/1993	Gu	384/106
4,451,163 A *	5/1984	Glaser		5,322,371 A	6/1994	Signoret	384/106
4,475,824 A	10/1984	Glaser	384/306	5,427,455 A *	6/1995	Bosley	
4,502,795 A	3/1985	Klaass	384/103	5,450,719 A	9/1995	Marsh	60/39.75
4,552,466 A	11/1985	Warren	384/103	5,498,083 A	3/1996	Brown	384/106
4,555,187 A	11/1985	Klaass	384/103	5,518,320 A	5/1996	Mohamed	384/103
4,616,388 A	10/1986	Soum	29/149.5 S	5,519,274 A	5/1996	Scharrer	310/90.5
4,743,126 A	5/1988	Soum	384/105	5,549,392 A *	8/1996	Anderson	384/104
4,795,274 A	1/1989	Gu	384/106	5,584,582 A	12/1996	Brown	384/106
4,808,070 A	2/1989	Fonda-Bonardi	415/110	5,634,723 A	6/1997	Agrawal	384/106
4,818,123 A	4/1989	Gu	384/106	5,658,079 A	8/1997	Struziak	384/106
4,896,975 A	1/1990	Bescoby	384/114	5,885,004 A	3/1999	Scharrer	384/99
4,950,089 A	8/1990	Jones	384/103	5,902,049 A	5/1999	Heshmat	384/106
4,961,122 A	10/1990	Sakai	360/107	5,911,510 A	6/1999	Saville	384/103
5,017,022 A	5/1991	Ruggles	384/100	5,911,511 A	6/1999	Saville	384/106
5,116,143 A	5/1992	Saville	384/106	5,915,841 A	6/1999	Weissert	384/104
5,129,739 A	7/1992	Asai	384/292	6,024,491 A	2/2000	Bak	384/106
5,228,785 A	7/1993	Saville	384/103				

\* cited by examiner

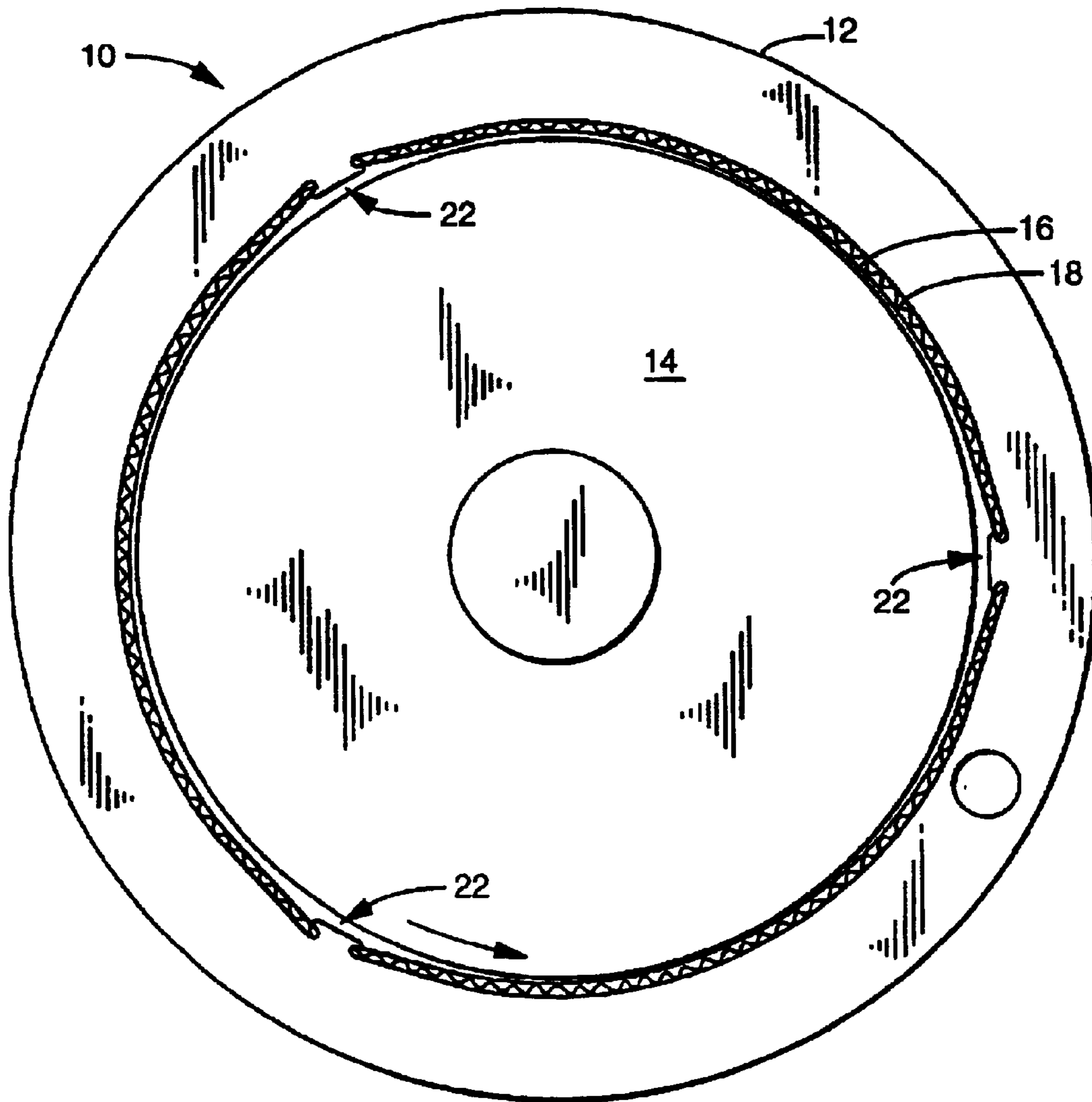


FIG. 1



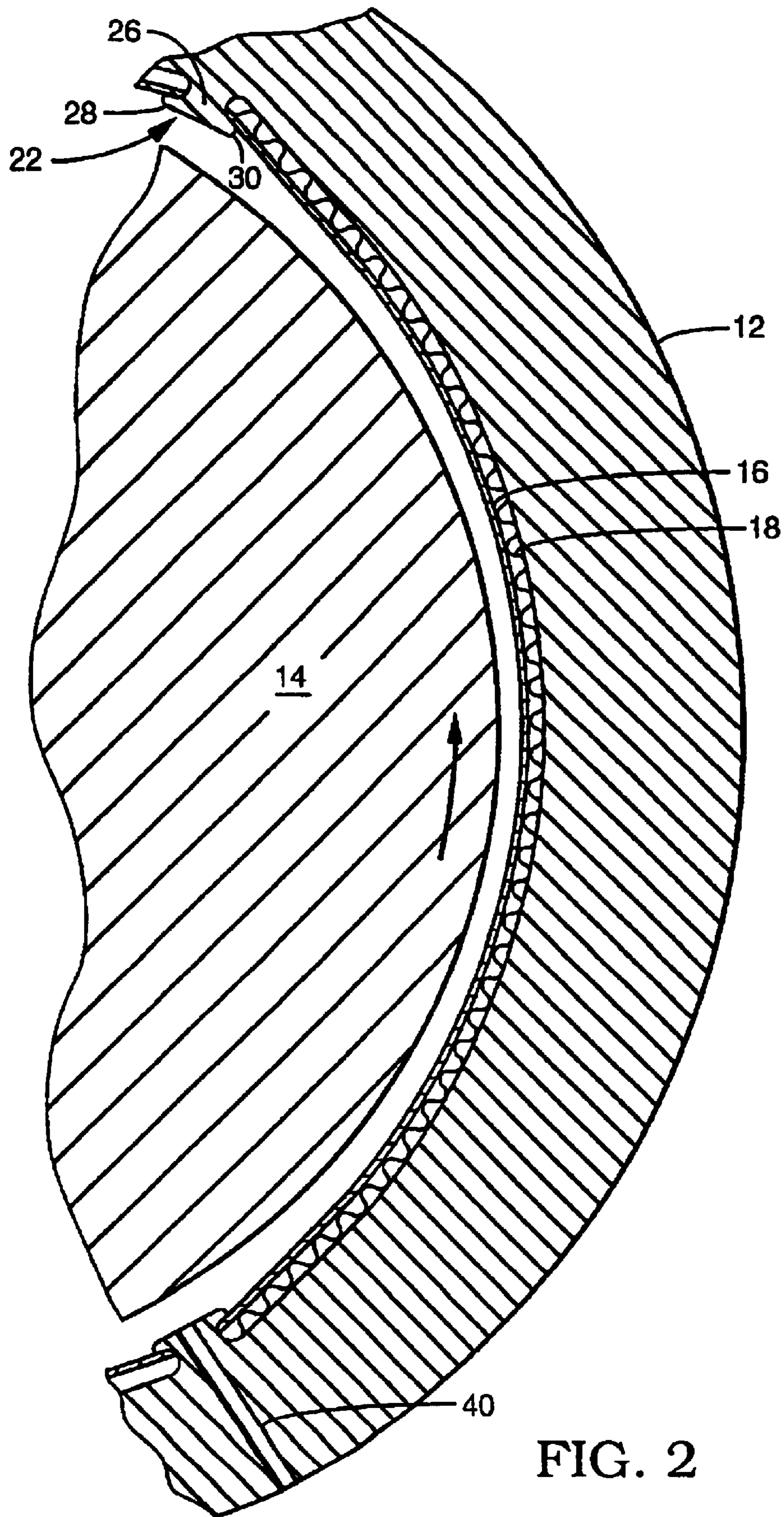
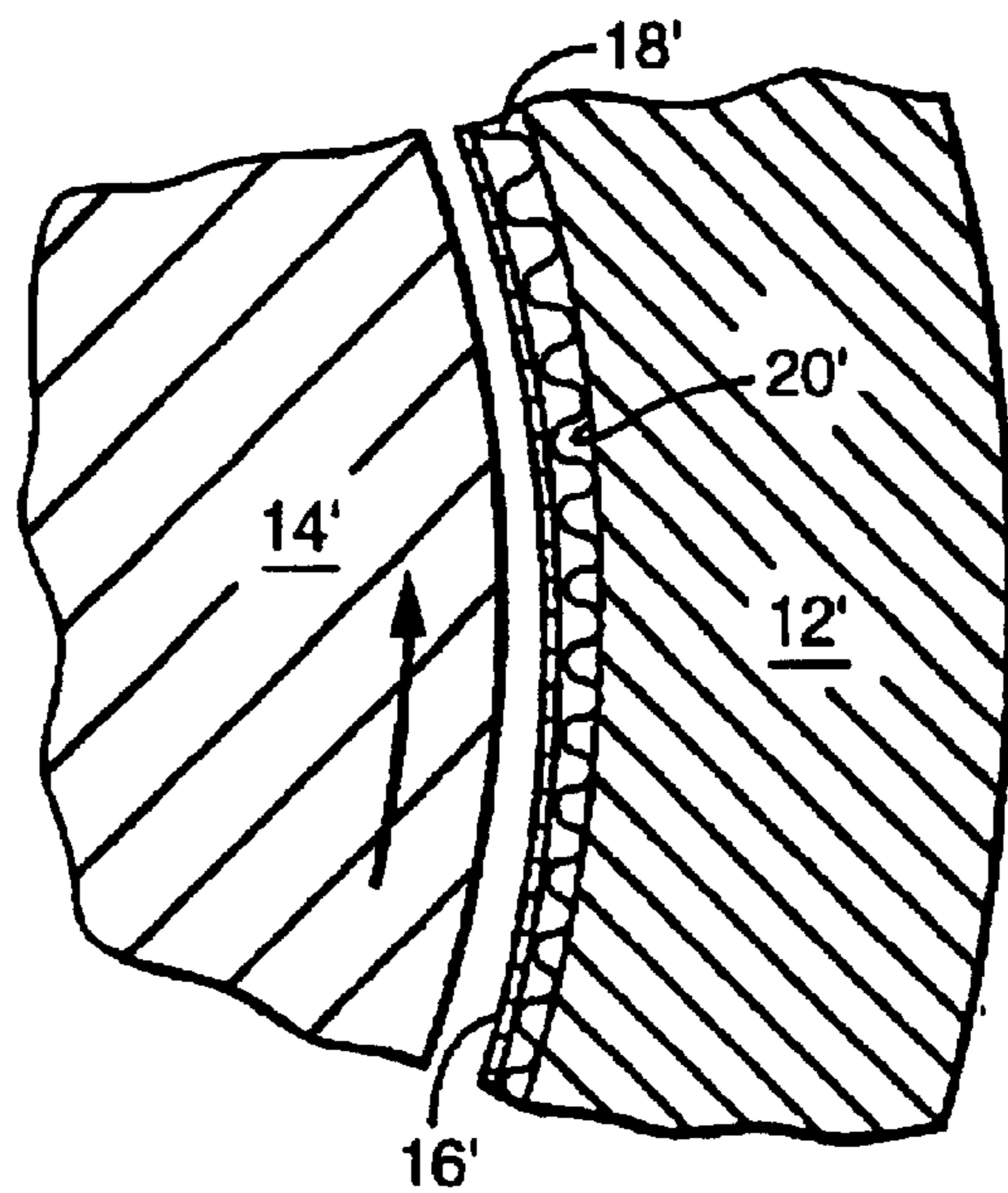
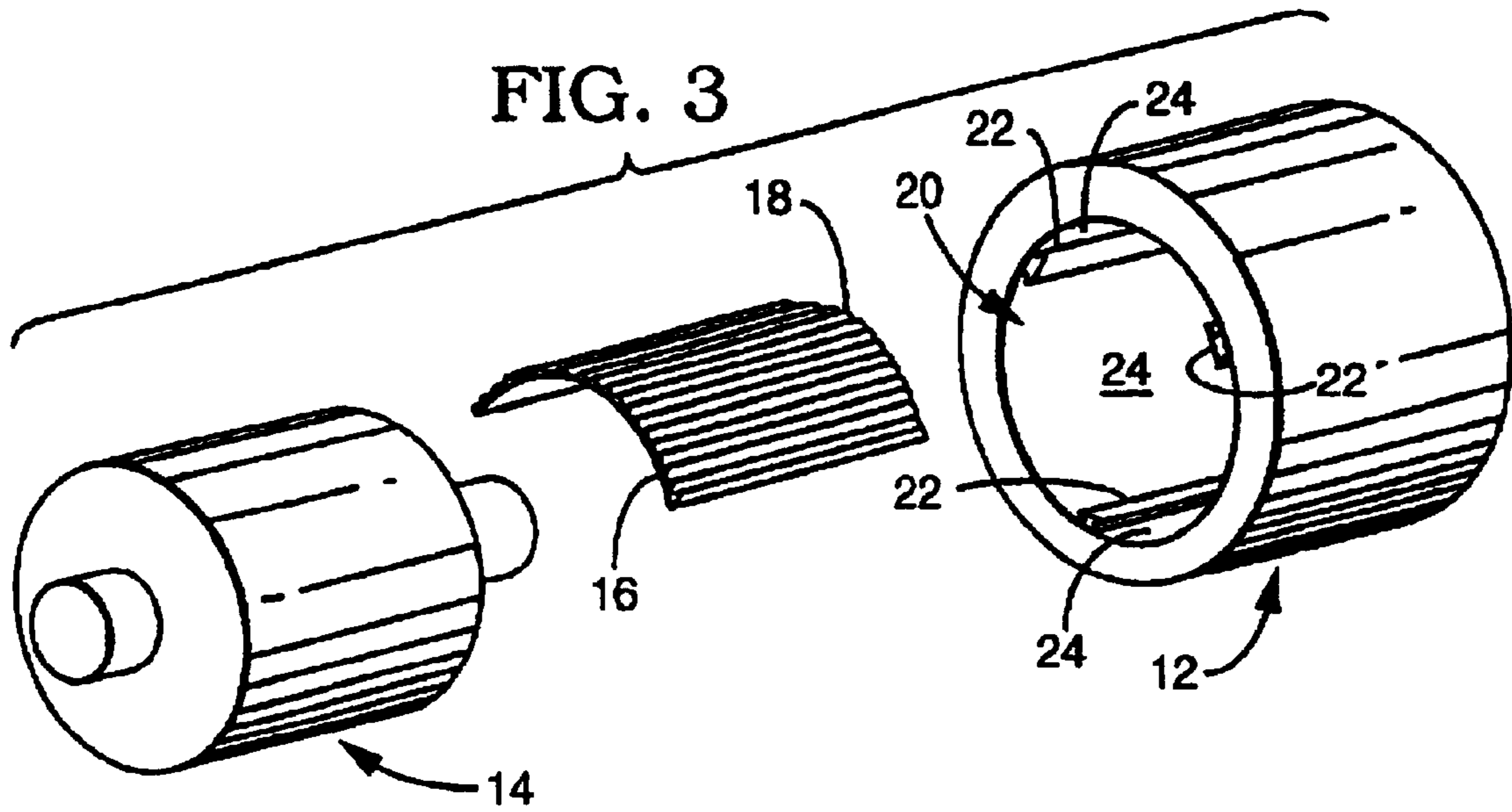


FIG. 2



**FIG. 4**



## COMPLIANT FOIL FLUID FILM RADIAL BEARING

Matter enclosed in heavy brackets [ ] appears in the original patent but forms no part of this reissue specification; matter printed in italics indicates the additions made by reissue.

### TECHNICAL FIELD

This invention relates to the general field of compliant foil fluid film bearings and more particularly to an improved multi segment compliant foil fluid film radial bearing.

### BACKGROUND OF THE INVENTION

Compliant foil fluid film radial bearings are currently being utilized in a variety of high speed rotor applications. These bearings are generally comprised of a bushing, a rotating element such as a rotor or shaft adapted to rotate within the bushing, non-rotating compliant fluid foil members mounted within the bushing and enclosing the rotating element, and non-rotating compliant spring foil members mounted within the bushing underneath the non-rotating compliant fluid foil members. The space between the rotating element and the bushing is filled with fluid (usually air) which envelops the foils. Conventionally, the compliant fluid foil elements are divided into a plurality of individual compliant foils to form a plurality of wedge shaped channels which converge in thickness in the direction of the rotation of the rotor.

The motion of the rotating element applies viscous drag forces to the fluid in the converging wedge channels. This results in increases in fluid pressure, especially near the trailing end of the wedge channels. If the rotating element moves toward the non-rotating element, the convergence angle of the wedge channel increases, causing the fluid pressure rise along the channel to increase. Conversely, if the rotating element moves away, the pressure rise along the wedge channel decreases. Thus, the fluid in the wedge channels exerts restoring forces on the rotating element that vary with and stabilize running clearances and prevent contact between the rotating and non-rotating elements of the bearing. Flexing and sliding of the foils causes coulomb damping of any axial or overturning motion of the rotating element of the bearing.

Owing to preload spring forces or gravity forces, the rotating element of the bearing is typically in physical contact with the fluid foil members of the bearing at low rotational speeds. This physical contact results in bearing wear. It is only when the rotor speed is above what is termed the lift-off/touch-down speed that the fluid dynamic forces generated in the wedge channels assure a running gap between the rotating and non-rotating elements.

Compliant foil fluid film radial bearings typically rely on backing springs to preload the fluid foils against the relatively movable rotating element so as to control foil position/nesting and to establish foil dynamic stability. The bearing starting torque (which should ideally be zero) is directly proportional to these preload forces. These preload forces also significantly increase the rotor speed at which the hydrodynamic effects in the wedge channels are strong enough to lift the rotating element of the bearing out of physical contact with the non-rotating members of the bearing. These preload forces and the high lift-off/touchdown speeds result in significant bearing wear each time the rotor is started or stopped.

Conventional compliant foil fluid film radial bearings operate with extremely small running clearances and moderate, as opposed to low, drag and power consumption. The clearances between the non-rotating fluid foil's converging channel ramp trailing ends and the rotating element are typically less than 100 micro-inches at operating conditions.

While most prior compliant foil fluid film radial bearings utilize a plurality of individual compliant foils and individual spring foils with a cylindrically bored bushing, there are instances where a single compliant foil and a single spring foil have been proposed with a cam shaped or lobed bushing. An example of this type of radial bearing can be found in U.S. Pat. No. 5,427,455 issued Jun. 27, 1995 to Robert W. Bosley, entitled "Compliant Foil Hydrodynamic Fluid Film Radial Bearing".

### SUMMARY OF THE INVENTION

The present invention is directed to a multi-segment radial bearing generally comprising a bushing, a shaft or rotor, a plurality of compliant foils, and a like plurality of foil undersprings.

The interior bore of the bushing includes a plurality of anti-rotation retainers that are equally spaced and extend the axial length of the interior bore, which may be either cylindrical or contoured. These generally T-shaped retainers divide the interior bore of the bushing into a like plurality of lobes, with each lobe having a compliant foil and foil underspring disposed therein between adjacent generally T-shaped retainers.

### BRIEF DESCRIPTION OF THE DRAWINGS

Having thus described the present invention in general terms, reference will now be made to the accompanying drawings in which:

FIG. 1 is an end view of the compliant foil fluid film radial bearing of the present invention;

FIG. 2 is an enlarged sectional view of a portion of the compliant foil fluid film radial bearing of FIG. 1;

FIG. 3 is an exploded perspective view of the compliant foil fluid radial bearing of FIGS. 1 and 2; and

FIG. 4 is an enlarged sectional view of a portion of an alternate compliant foil fluid film radial bearing of the present invention.

### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Illustrated in FIGS. 1-3 is the compliant foil fluid film radial bearing 10 of the present invention. This multi-segment radial bearing 10 generally comprises a bushing 12, a shaft or rotor 14, a plurality of compliant foils 16 (shown as three), and a like plurality of foil undersprings 18.

The interior bore 20 of the bushing 12 includes a plurality of anti-rotation devices or retainers 22 (shown as three, generally T-shaped retainers) which are equally spaced and extend the axial length of the interior bore 20. The retainers 22 divide the interior bore 20 of the bushing 12 into a like plurality of lobes 24. The compliant foils 16 and the foil undersprings 18 are disposed in each lobe 24 between adjacent generally T-shaped retainers 22.

While the shaft or rotor 14 is cylindrical, the interior bore 20 of the bushing 12 may or may not be cylindrical. If the interior bore 20 is not cylindrical, the generally T-shaped retainers 22 will divide the interior bore 20 into a plurality of shaped or contoured lobes 24.



On the other hand, if the interior bore 20' of the bushing 12' is cylindrical, the generally T-shaped retainers will divide the interior bore 20' into a plurality of equally spaced arc segments of a cylinder and the converging wedges will be formed via the compliant foil 16' and/or the foil undersprings 18' as generally shown in FIG. 4. The height of the foil undersprings 18' may increase from the leading edge of the compliant foil 16' to the trailing edge of the compliant foil 16' in order to form the converging wedge. Alternately, the stiffness of the foil underspring can vary from the leading edge to the trailing edge to produce the same result, or shims of varying height can be positioned underneath the foil underspring.

The generally T-shaped retainers 22 may be symmetrical or conform to the leading edge and trailing edge of the contoured lobes 24. Non-symmetrical T-shaped retainers 22 will retain the foils 16 at different radii from the geometric center with the leading edge of the foils retained at a greater geometric radius than the trailing edge.

The generally T-shaped retainers 22 are formed as part of the contoured interior bore 20 of the bushing 12 with the base 26 thereof extending radially inwards from the interior bore 20. The cross piece of the generally T-shaped retainers 22 has a leading edge 30 and a trailing edge 28, with the leading edge 30 having a greater thickness or width than the trailing edge 28 as best shown in FIG. 2. Since the inner diameter of the cross piece is the same at both the leading edge 30 and the trailing edge 28, this provides a greater diameter or depth for the trailing edge of the contoured lobe 24 which is under the thicker leading edge 30 of the generally T-shaped retainers 22.

The contoured surface of the lobes 24 can be derived from one or a series of radii of a defined profile. For example, in the case of three (3) foil segments, the contour of the lobes 24 in which the compliant foils 16 and underspring foils 18 rest upon can be a multiple of, or a fraction of, a specific number. Also, an appropriate contour can be constructed from vertices of an equilateral triangle of specific length side with circles formed from these vertices intersecting to give the contoured surface which may be machined, ground, EDM'ed or broached.

While a preferred embodiment of the compliant foil fluid film radial bearing 10 is shown with three (3) compliant foil segments 16 and three (3) foil undersprings 18, a greater number of compliant foils segments 16 and foil undersprings 18 can be utilized. For example, five (5) or more segments may be equally appropriate for the compliant foil fluid film radial bearing 10.

The compliant foils 16 and foil undersprings 18 are trapped and held between adjacent T-shaped retainers 22. The compliant foils 16 would normally have a preformed arcuate shape as shown in FIG. 3. The foil undersprings 18 may have a preferred radius or simply be a rectangular sheet. In either case, the compliant foils 16 and foil undersprings 18 are axially inserted, either separately or together, into the interior bore 20 of bushing 12 between adjacent T-shaped retainers 22. Also, while the foil undersprings 18 are illustrated as a wavy springform, any conventional bearing underspring can be utilized, including the spring described in U.S. Pat. No. 5,427,455. The underspring 18 may have variable spring rates or tapered heights from the leading edge to the trailing edge as shown in FIG. 4.

When trapped between adjacent T-shaped retainers 22, the compliant foils 16 would be pre-loaded in compression between the adjacent T-shaped retainers 22. In some instances, however, the compliant foils 16 may be merely

retained in position between adjacent T-shaped retainers 22 without preloading. The foil undersprings 18 would, however, normally be pre-loaded in compression between adjacent T-shaped retainers 22. The foil underspring 18 may contribute to the contoured shape of the compliant foils 16. In most instances, the compliant foils 16 and foil underspring 18 would be separate and would be capable of sliding movement therebetween. There may, however, be instances where the compliant foils 16 and foil underspring 18 will be joined together into a single assembly.

The greater thickness of the leading edge 30 of the T-shaped retainer crosspiece produces a greater depth at the trailing edge of the compliant foils 16 to serve as a means to pull the compliant foils 16 away from the trailing edge of the converging wedge. The trailing edge 28 of the T-shaped retainers 22 has less thickness thereby allowing the leading edge of the foils to rest closer to the shaft, which will facilitate development of the fluid film wedges.

Also, while the compliant foil fluid film radial bearing 10 can function hydrodynamically, by providing an external source of fluid under pressure, a hydrostatic aspect can be realized. The external pressurized fluid may be introduced axially at the ends of the bushing or through radial holes 40 in the bushing 40 axially along and through the T-shaped retainers 22. The hydrostatic aspect will provide cooling fluid to the interior bore 20 of the bushing 12, which will increase bearing life especially in a gas turbine application. Hydrostatic operation will also increase the load capacity of the bearing.

The radial bearing 10 of the present invention allows for automation by mass production and the components can easily be assembled by hand. The compliant foils 16 may be stamped or fine blanked and there is no requirement for spot welding or chemical etching. The large clearances between the compliant foils 16 and the shaft 14 at the T-shaped retainers 22 allow improved cooling of the shaft 14 and compliant foils 16. The design can accommodate a variety of underspring types and the T-shaped retainers 22 permit more design flexibility with respect to the shaft, foil and spring interaction.

While specific embodiments of the invention have been illustrated and described, it is to be understood that there are provided by way of example only and that the invention is not to be construed as being limited thereto but only by the proper scope of the following claims.

What I claim is:

1. A compliant foil fluid film radial bearing comprising:
  - a bushing having an interior bore including a plurality of equally spaced generally T-shaped retainers axially extending in said interior bore and a like plurality of lobes between adjacent generally T-shaped retainers;
  - a shaft rotatably supported within said interior bore of said bushing;
  - a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing between adjacent generally T-shaped retainers; and
  - a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils between adjacent generally T-shaped retainers.

2. The compliant foil fluid film radial bearing of claim 1 wherein said interior bore is cylindrical and said individual compliant foils and said individual foil undersprings beneath said individual complaint foils establish a converging wedge between adjacent generally T-shaped retainers.

3. The compliant foil fluid film radial bearing of claim 1 wherein said interior bore is non-cylindrical, generally con-



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toured lobes are formed between adjacent generally T-shaped retainers, and said individual compliant foils and said individual foil undersprings generally conform to the shape of said contoured lobes to establish a converging wedge.

4. A compliant foil fluid film radial bearing comprising:  
 a bushing having a cylindrical interior bore including a plurality of equally spaced retainers axially extending into said interior bore and a like plurality of arc segments between adjacent retainers;  
 a shaft rotatably supported within said interior bore of said bushing;  
 a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing in each arc segment between adjacent retainers; and  
 a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils in each arc segment between adjacent retainers, the radial height of said foil undersprings increasing from its leading edge to its trailing edge to establish a converging wedge on the surface of said compliant foil facing said shaft.

5. The compliant foil fluid film radial bearing of claim 4 wherein said retainers are generally T-shaped.

6. The compliant foil fluid film radial bearing of claim 5 wherein said generally T-shaped retainers are symmetrical.

7. The compliant foil fluid film radial bearing of claim 5 wherein said generally T-shaped retainers are asymmetrical.

8. The compliant foil fluid film radial bearing of claim 7 wherein said generally T-shaped asymmetrical retainers have a crosspiece with a leading edge and a trailing edge, with the leading edge thicker than the trailing edge.

9. The compliant foil fluid film radial bearing of claim 4 wherein the radial stiffness of said foil undersprings increases from its leading edge to its trailing edge.

10. The compliant foil fluid film radial bearing of claim 9 wherein said bearing is hydrodynamic.

11. The compliant foil fluid film radial bearing of claim 9 wherein said retainers are generally T-shaped.

12. The compliant foil fluid film radial bearing of claim 9 wherein said bearing is hydrostatic.

13. A compliant foil fluid film radial bearing comprising:  
 a bushing having a non-cylindrical interior bore including a plurality of equally spaced retainers axially extending into said interior bore and a like plurality of contoured lobes between adjacent retainers;  
 a shaft rotatably supported within said interior bore of said bushing;  
 a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing in each contoured lobe between adjacent retainers; and  
 a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils in each contoured lobe between adjacent retainers, the contour of each lobe between adjacent retainers establishing a converging wedge on the surface of said compliant foil facing said shaft.

14. The compliant foil fluid film radial bearing of claim [13] 56 wherein said generally T-shaped retainers are symmetrical.

15. The compliant foil fluid film radial bearing of claim [14] 16 wherein said generally T-shaped asymmetrical retainers have a crosspiece with a leading edge and a trailing edge, with the leading edge thicker than the trailing edge.

16. The compliant foil fluid film radial bearing of claim [13] 56 wherein said generally T-shaped retainers are [symmetrical] asymmetrical.

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17. The compliant foil fluid film radial bearing of claim [13] 56 wherein said generally T-shaped retainers include radial openings to provide cooling flow to said interior bore of said bushing.

18. The compliant foil fluid film radial bearing of claim 13 and in addition means to provide cooling flow axially into said interior bore of said bushing.

19. A compliant foil fluid film radial bearing comprising:  
 a bushing having an interior bore including a plurality of generally T-shaped retainers axially extending in said interior bore;  
 a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing between adjacent generally T-shaped retainers; and  
 a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils between adjacent generally T-shaped retainers.

20. The compliant foil fluid film radial bearing of claim 19 wherein said interior bore is cylindrical and said individual compliant foils and said individual foil undersprings beneath said individual compliant foils establish a converging wedge between adjacent generally T-shaped retainers.

21. The compliant foil fluid film radial bearing of claim 19 wherein said interior bore is non-cylindrical, generally contoured lobes are formed between adjacent generally T-shaped retainers, and said individual compliant foils and said individual foil undersprings generally conform to the shape of said contoured lobes to establish a converging wedge.

22. The compliant foil fluid film radial bearing of claim 19 wherein said bearing is hydrodynamic.

23. The compliant foil fluid film radial bearing of claim 19 wherein said bearing is hydrostatic.

24. The compliant foil fluid film radial bearing of claim 19 wherein said generally T-shaped retainers include radial openings to provide cooling flow to said interior bore of said bushing.

25. The compliant foil fluid film radial bearing of claim 19 further comprising:

means to provide cooling flow axially into said interior bore of said bushing.

26. The compliant foil fluid film radial bearing of claim 13 wherein said retainers are generally T-shaped retainers.

27. A compliant foil fluid film radial bearing comprising:  
 a bushing having a cylindrical interior bore including a plurality of retainers axially extending into said interior bore;  
 a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing between adjacent retainers; and  
 a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils between adjacent retainers, the radial height of said foil undersprings increasing from its leading edge to its trailing edge to establish a converging wedge of said compliant foil.

28. The compliant foil fluid film radial bearing of claim 27 wherein said retainers are generally T-shaped.

29. The compliant foil fluid film radial bearing of claim 28 wherein said generally T-shaped retainers are asymmetrical.

30. The compliant foil fluid film radial bearing of claim 29 wherein said generally T-shaped asymmetrical retainers have a crosspiece with a leading edge and a trailing edge, with the leading edge thicker than the trailing edge.

31. The compliant foil fluid film radial bearing of claim 28 wherein said generally T-shaped retainers are symmetrical.



32. The compliant foil fluid film radial bearing of claim 28 wherein said generally T-shaped retainers include radial openings to provide cooling flow to said interior bore of said bushing.

33. The compliant foil fluid film radial bearing of claim 27 wherein the radial stiffness of said foil undersprings increases from its leading edge to its trailing edge.

34. The compliant foil fluid film radial bearing of claim 33 wherein said bearing is hydrodynamic.

35. The compliant foil fluid film radial bearing of claim 33 wherein said bearing is hydrostatic.

36. The compliant foil fluid film radial bearing of claim 27 further comprising:

means to provide cooling flow axially into said interior bore of said bushing.

37. A compliant foil fluid radial bearing comprising:

a bushing having a non-cylindrical interior bore including a plurality of retainers axially extending into said interior bore;

a plurality of compliant foils, with an individual compliant foil disposed in said interior bore of said bushing between adjacent retainers; and

a plurality of foil undersprings, with an underspring disposed beneath each of said compliant foils between adjacent retainers, the contour of the interior bore between adjacent retainers establishing a converging wedge on the surface of said compliant foil.

38. The compliant foil fluid radial bearing of claim 37 wherein said retainers are generally T-shaped.

39. The compliant foil fluid film radial bearing of claim 38 wherein said generally T-shaped retainers are asymmetrical.

40. The compliant foil fluid film radial bearing of claim 39 wherein said generally T-shaped asymmetrical retainers have a crosspiece with a leading edge and a trailing edge, with the leading edge thicker than the trailing edge.

41. The compliant foil fluid film radial bearing of claim 38 wherein said generally T-shaped retainers are symmetrical.

42. The compliant foil fluid film radial bearing of claim 38 wherein said bearing is hydrostatic.

43. The compliant foil fluid film radial bearing of claim 38 wherein said generally T-shaped retainers include radial openings to provide cooling flow to said interior bore of said bushing.

44. The compliant foil fluid film radial bearing of claim 38 wherein said bearing is hydrodynamic.

45. The compliant foil fluid film radial bearing of claim 37 further comprising:

means to provide cooling flow axially into said interior bore of said bushing.

46. A radial bearing bushing comprising:

a bushing having an interior bore;

one or more retainer bases axially extending into the interior bore;

one or more leading edges attached to each of the one or more retainer bases for retaining a compliant foil; and

one or more trailing edges attached to each of the one or more retainer bases for retaining a compliant foil.

47. The radial bearing of claim 46 wherein the interior bore is cylindrical.

48. The radial bearing of claim 46 wherein the interior bore is non-cylindrical.

49. The radial bearing of claim 46 wherein one or more of the one or more retainer bases include radial openings to provide cooling flow to said interior bore of said bushing.

50. A compliant foil radial bearing comprising:

a bushing having an interior bore including one or more retainer bases axially extending into the interior bore;

one or more compliant foils;

one or more leading edges attached to each of the one or more retainer bases for retaining a compliant foil trailing edge;

one or more trailing edges attached to each of the one or more retainer bases for retaining a compliant foil leading edge; and

one or more foil undersprings, each underspring disposed beneath a compliant foil.

51. The compliant foil radial bearing of claim 50 wherein the interior bore is non-cylindrical, generally contoured lobes are formed between adjacent retainer bases, and an individual compliant foil and underspring beneath the individual compliant foil generally conform to the shape of a contoured lobe to establish a converging wedge.

52. The compliant foil radial bearing of claim 50 wherein said bearing is hydrodynamic.

53. The compliant foil radial bearing of claim 50 wherein said bearing is hydrostatic.

54. The compliant foil radial bearing of claim 50 wherein the one or more retainer bases include radial openings to provide cooling flow to the interior bore of said bushing.

55. The compliant foil radial bearing of claim 50 further comprising:

means to provide cooling flow axially into the interior bore of the bushing.

56. The compliant foil radial bearing of claim 50 wherein the interior bore is cylindrical and an individual compliant foil and underspring beneath the individual compliant foil form a converging wedge.

UNITED STATES PATENT AND TRADEMARK OFFICE  
**CERTIFICATE OF CORRECTION**

PATENT NO. : RE 39,190 E  
APPLICATION NO. : 09/895568  
DATED : July 18, 2006  
INVENTOR(S) : Weissert

Page 1 of 1

It is certified that error appears in the above-identified patent and that said Letters Patent is hereby corrected as shown below:

Column 8, line 24, replace "tailing" with --trailing--.

Signed and Sealed this

Fifth Day of December, 2006

A handwritten signature in black ink on a dotted background. The signature reads "Jon W. Dudas" in a cursive style.

JON W. DUDAS

*Director of the United States Patent and Trademark Office*