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Gharagozlu

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(54) **DEVICE AND METHOD FOR ORE-CRUSHING WITH A SPRING DEVICE**

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B02C 7/14 (2006.01)
(Continued)

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CPC ***B02C 7/14*** (2013.01); ***B02C 7/06*** (2013.01); ***B02C 19/0012*** (2013.01); ***B02C 21/02*** (2013.01)

(58) **Field of Classification Search**
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(Continued)

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(57) **ABSTRACT**

The invention regards to a device for comminuting ore and/or slag, which comprises an ore feed unit for feeding ore to be comminuted to a first comminuting means, the first comminuting means being composed of at least two comminuting elements that can be moved relative to each other, which elements form at least one comminuting space for the ore to be comminuted with each other such that, by a relative movement in the form of a rotation around the rotational axis of at least one of the two comminuting elements, the ore to be comminuted is pulverized in that one or more accelerating elements, in particular protrusions, are provided on at least one of the comminuting elements, said accelerating elements being arranged in particular on the end face of one of the two comminuting elements and accelerating and
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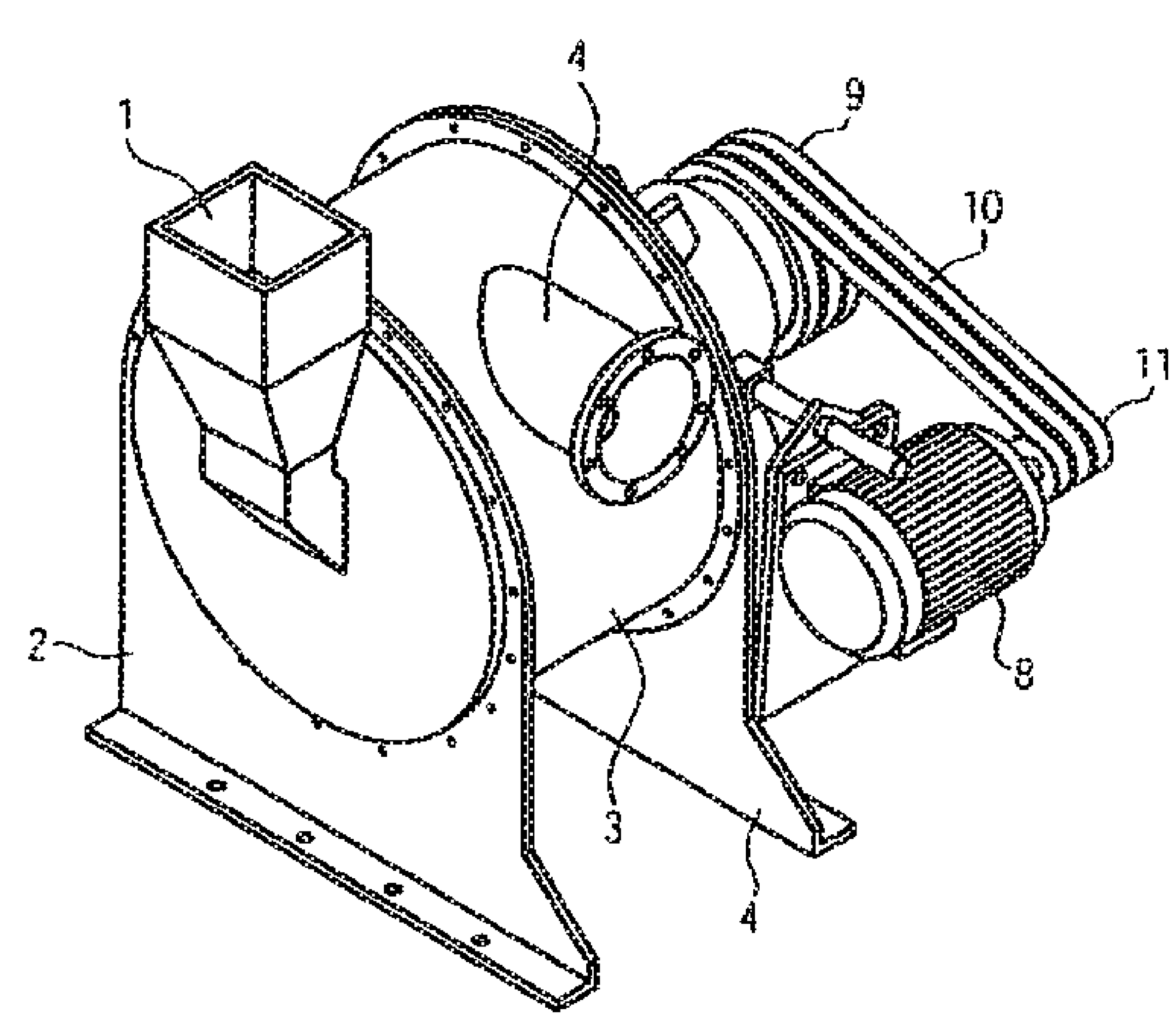

comminuting the ore to be comminuted by the rotation of one of the two comminuting elements,
and wherein at least one of the two comminuting elements comprises a functional connection with a spring means, wherein the spring means is formed in such a way, that it mounts the comminuting element being in functional connection with variably in the direction of the other comminuting element.

8 Claims, 21 Drawing Sheets

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B02C 7/06 (2006.01)
B02C 19/00 (2006.01)
B02C 21/02 (2006.01)

(58) **Field of Classification Search**
USPC 241/261.3, 261.2
See application file for complete search history.

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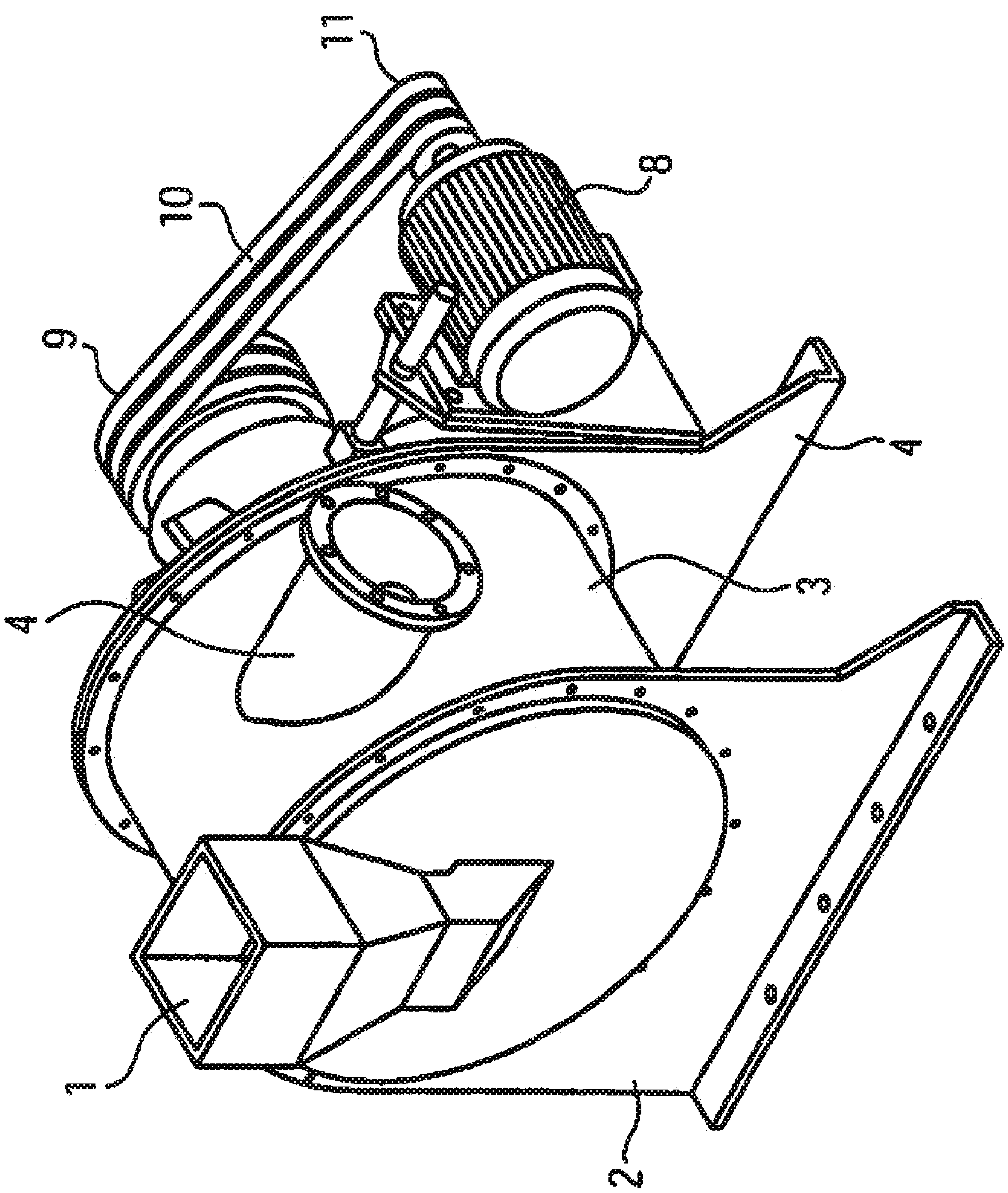

Fig. 1

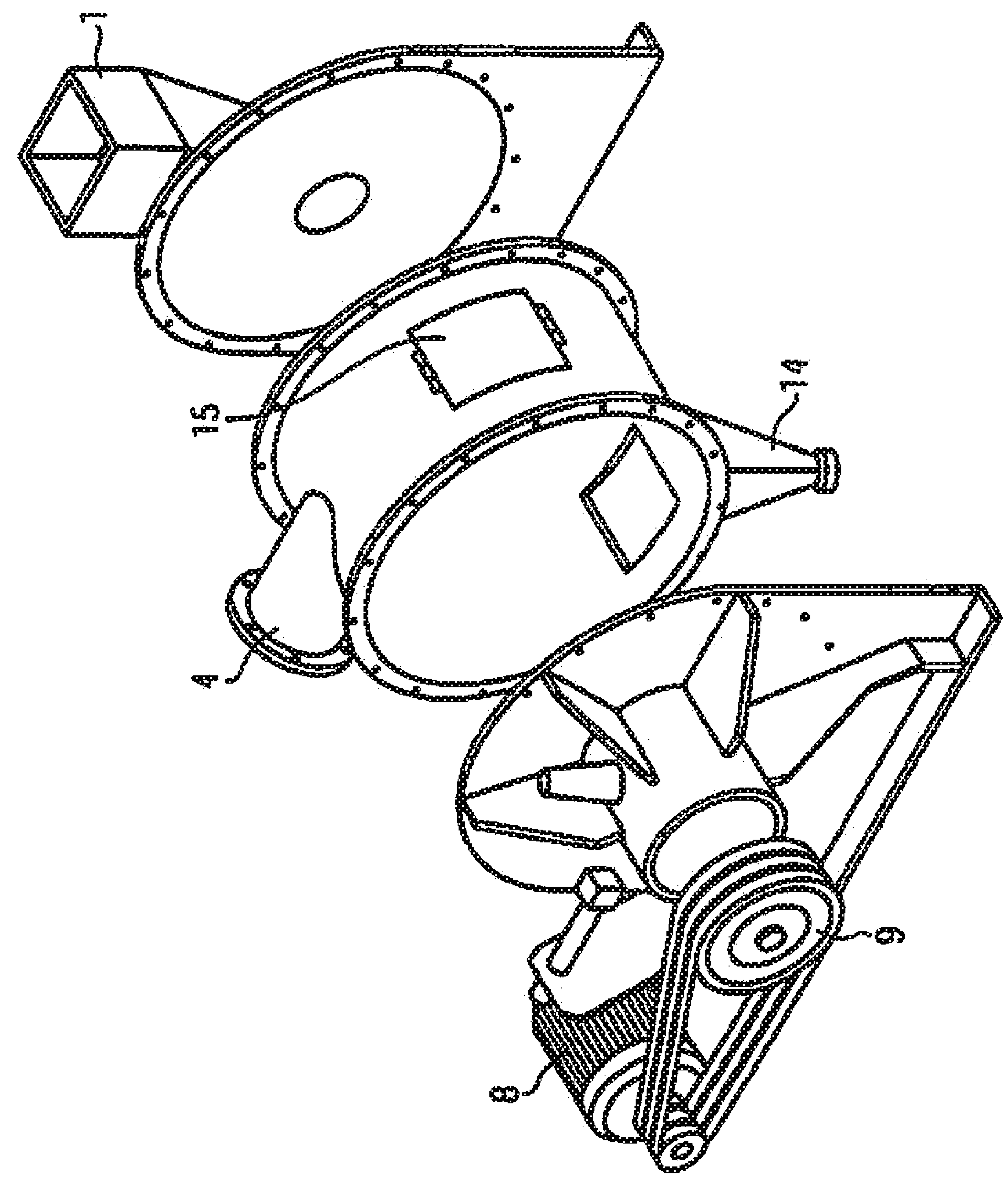

Fig. 2

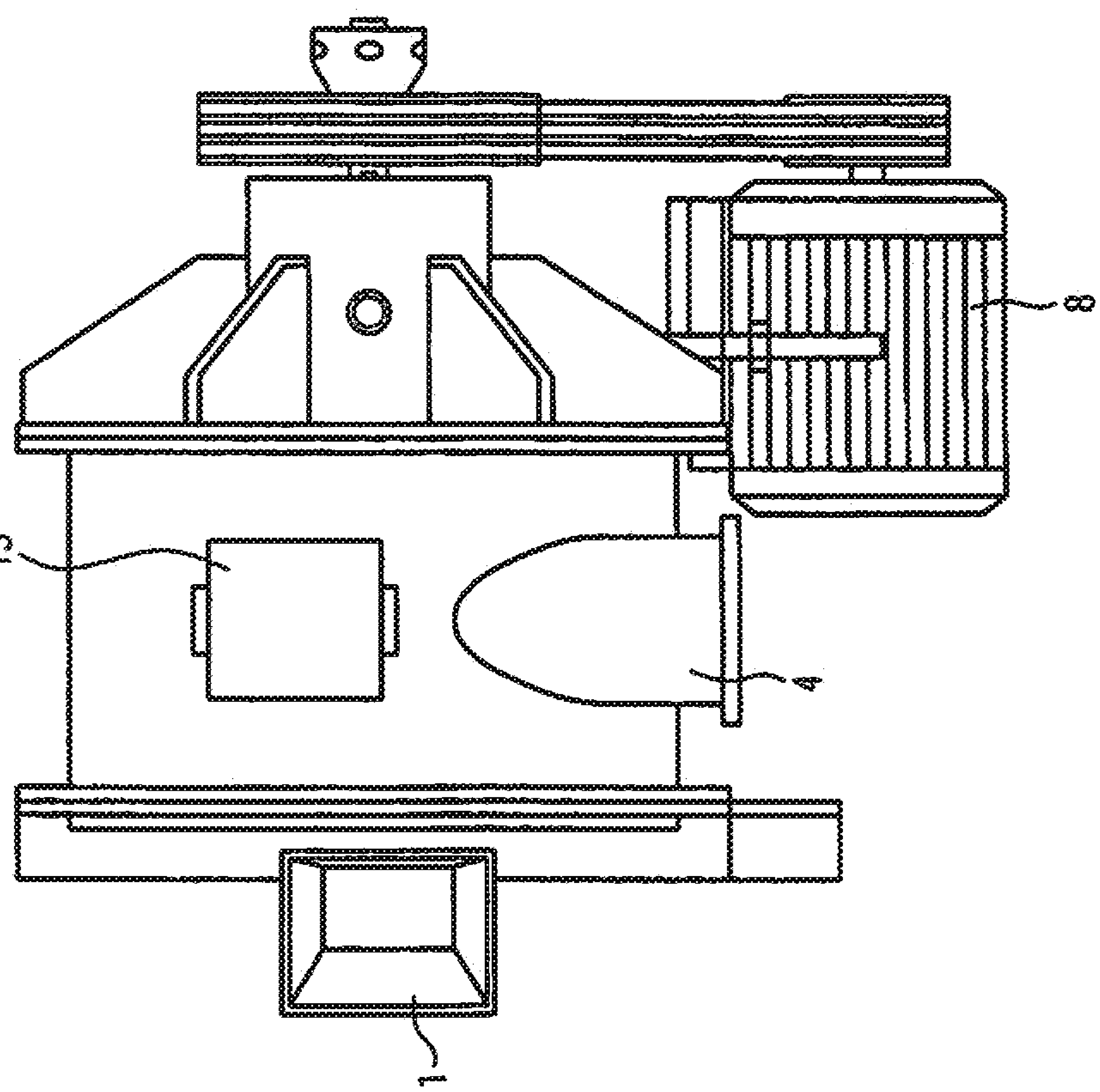

Fig. 3

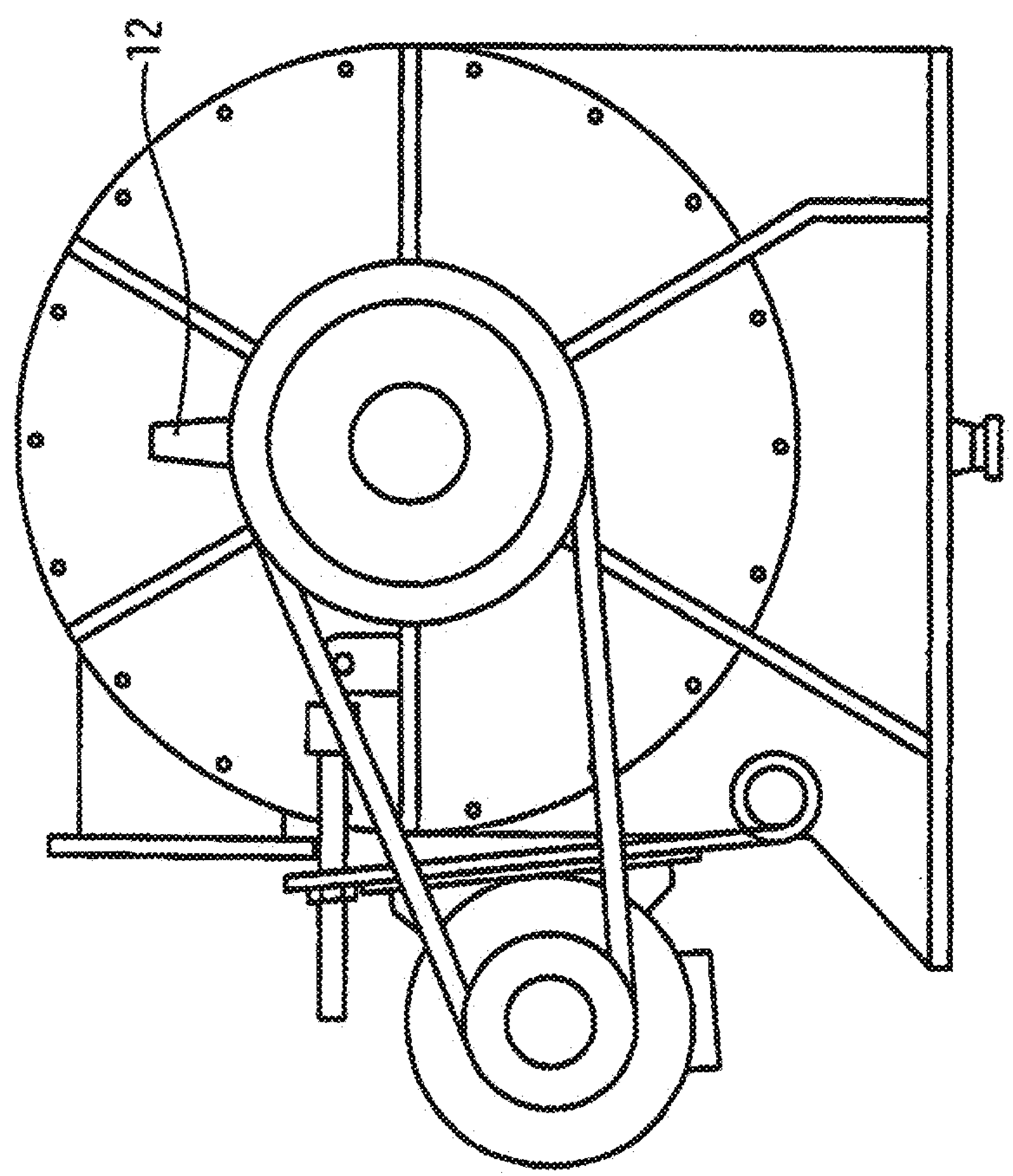

Fig. 4

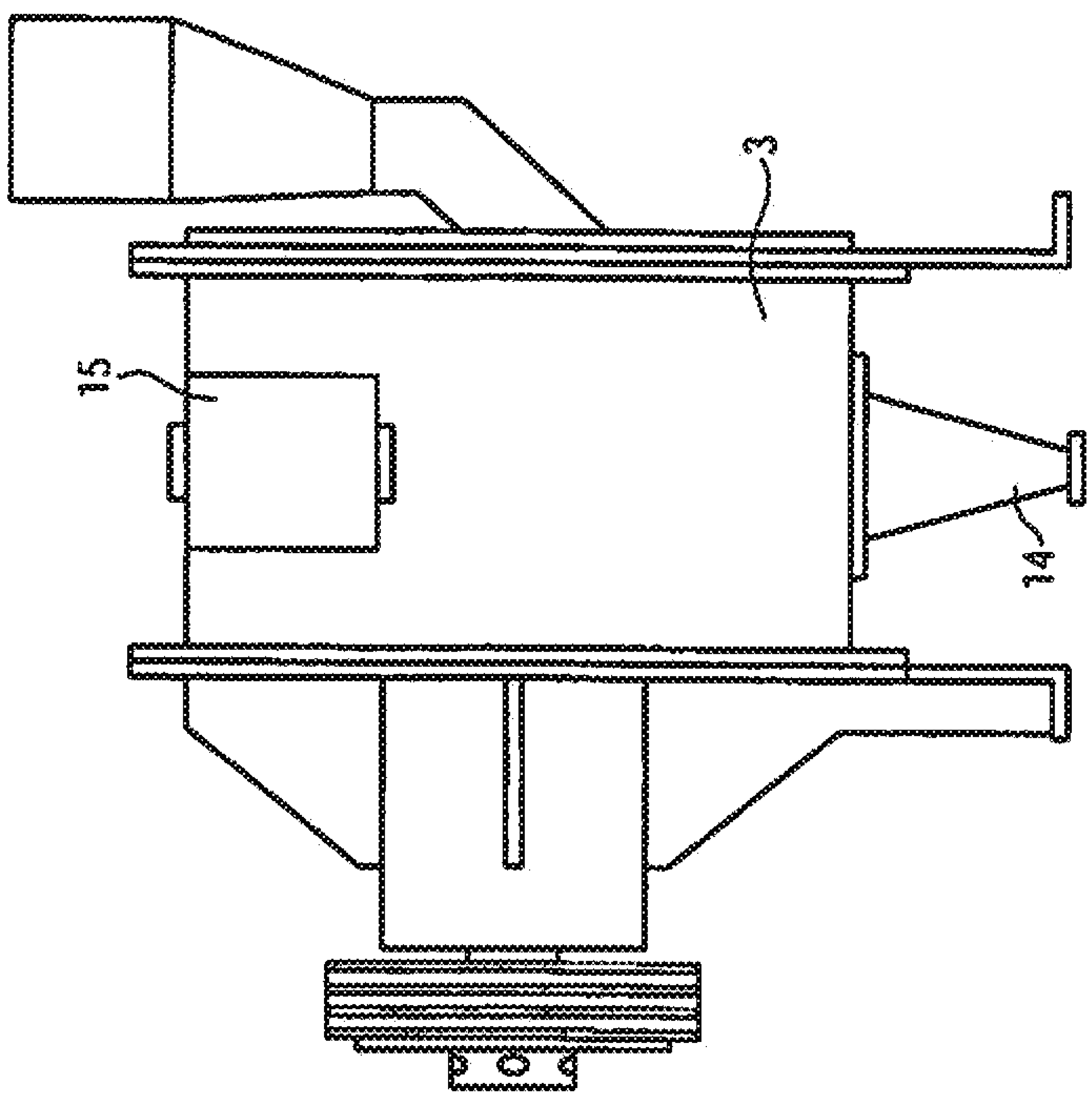

Fig. 5

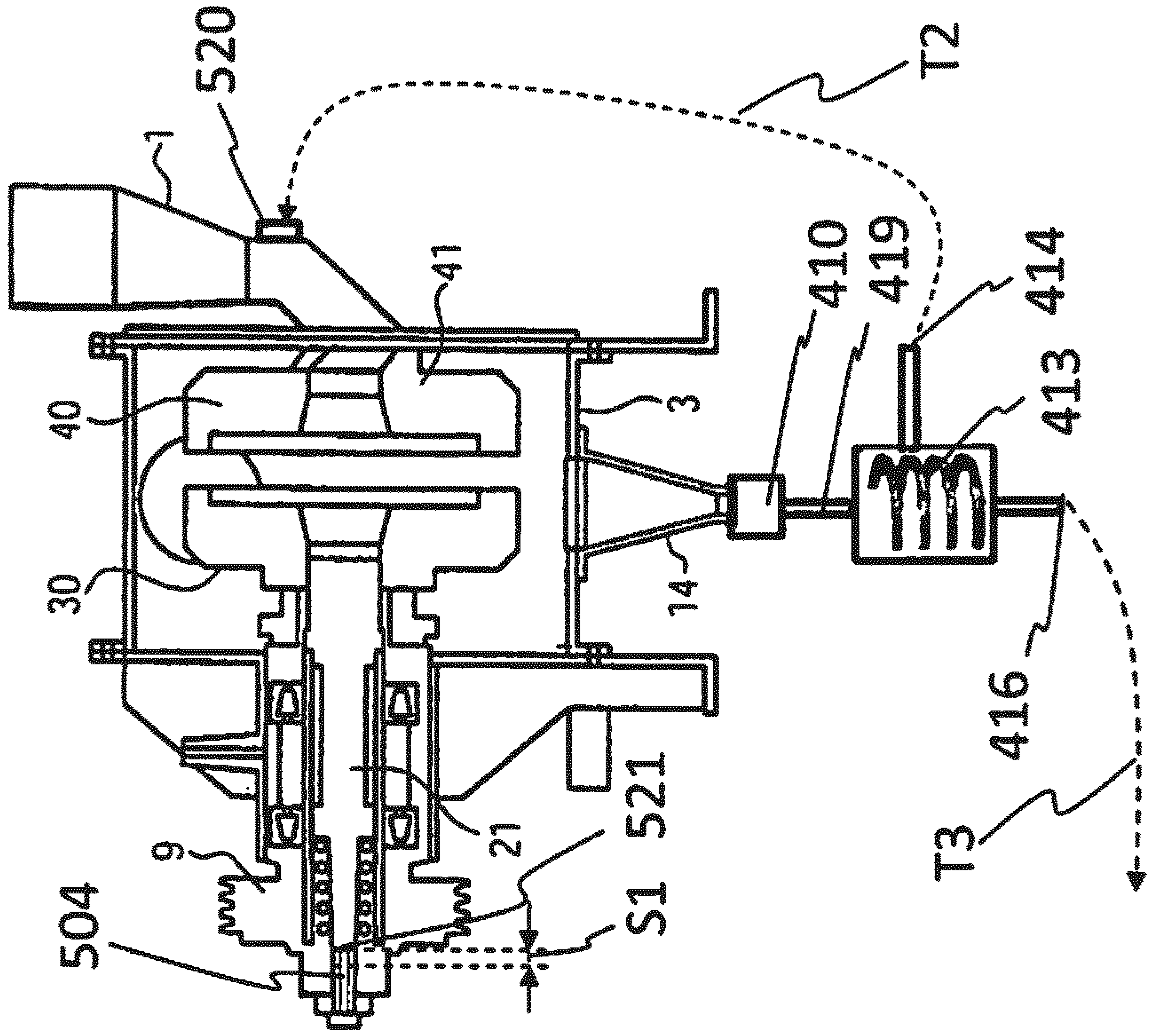

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Fig. 6b

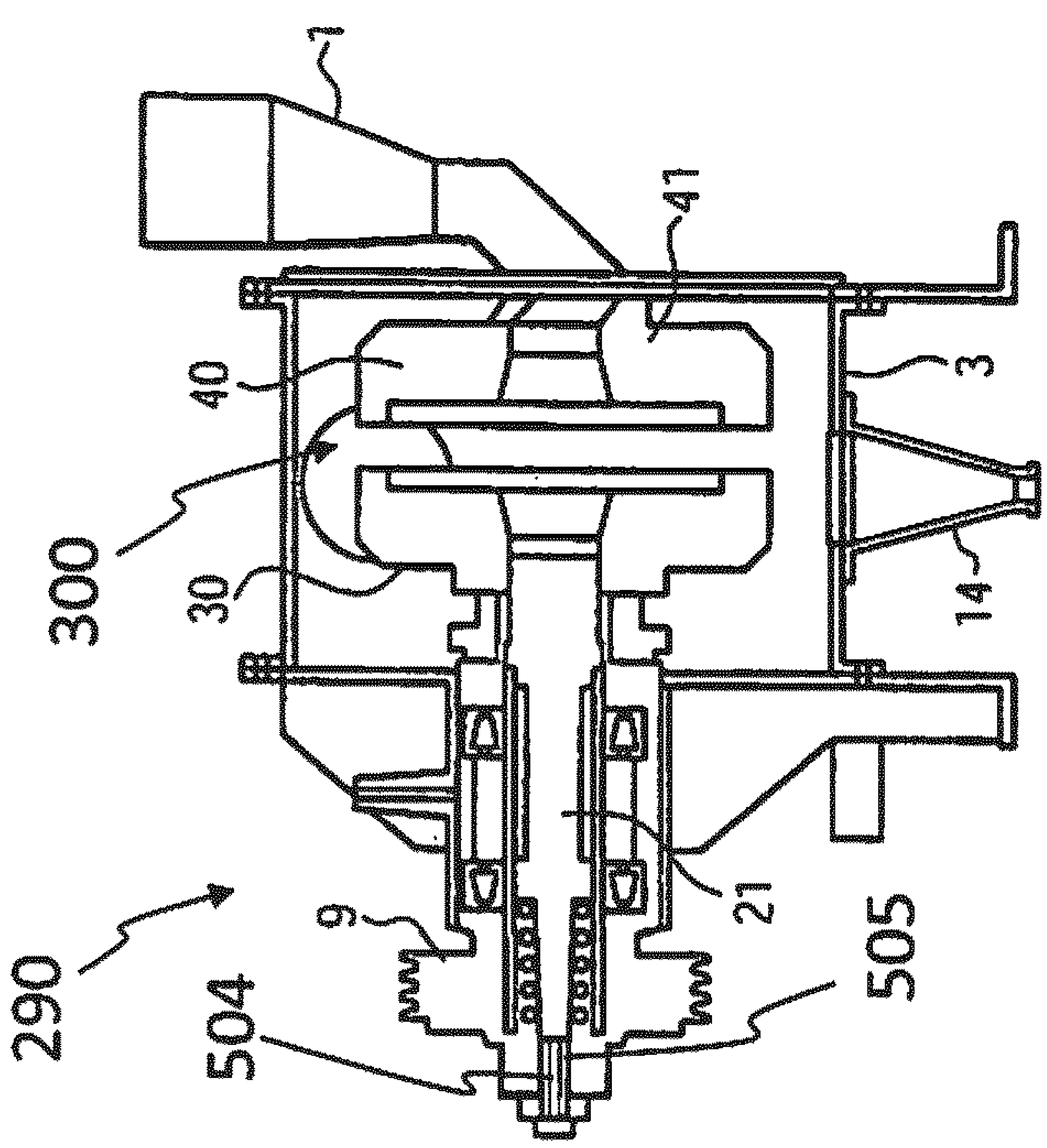

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Fig. 6a

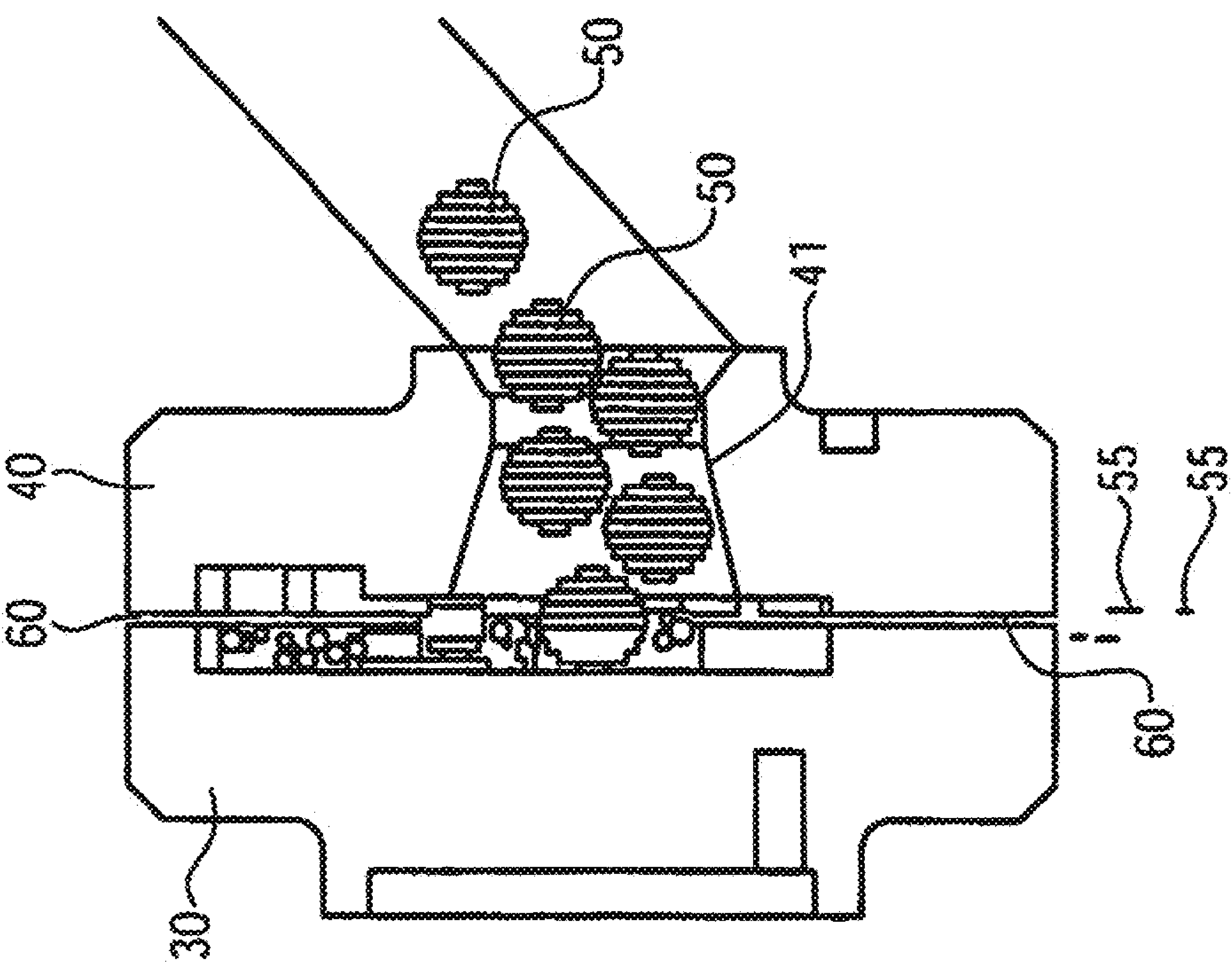

Fig. 7

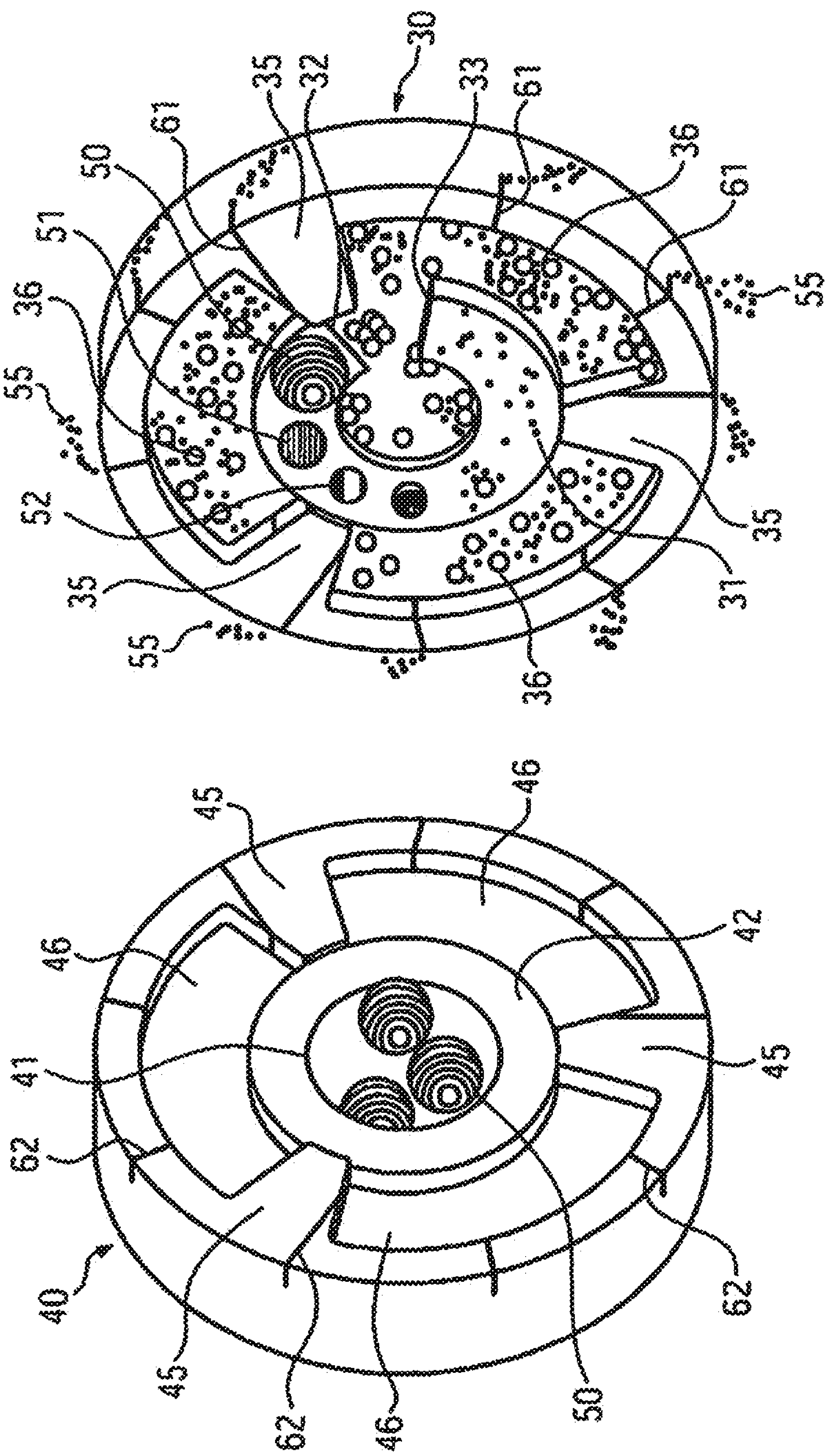

Fig. 8

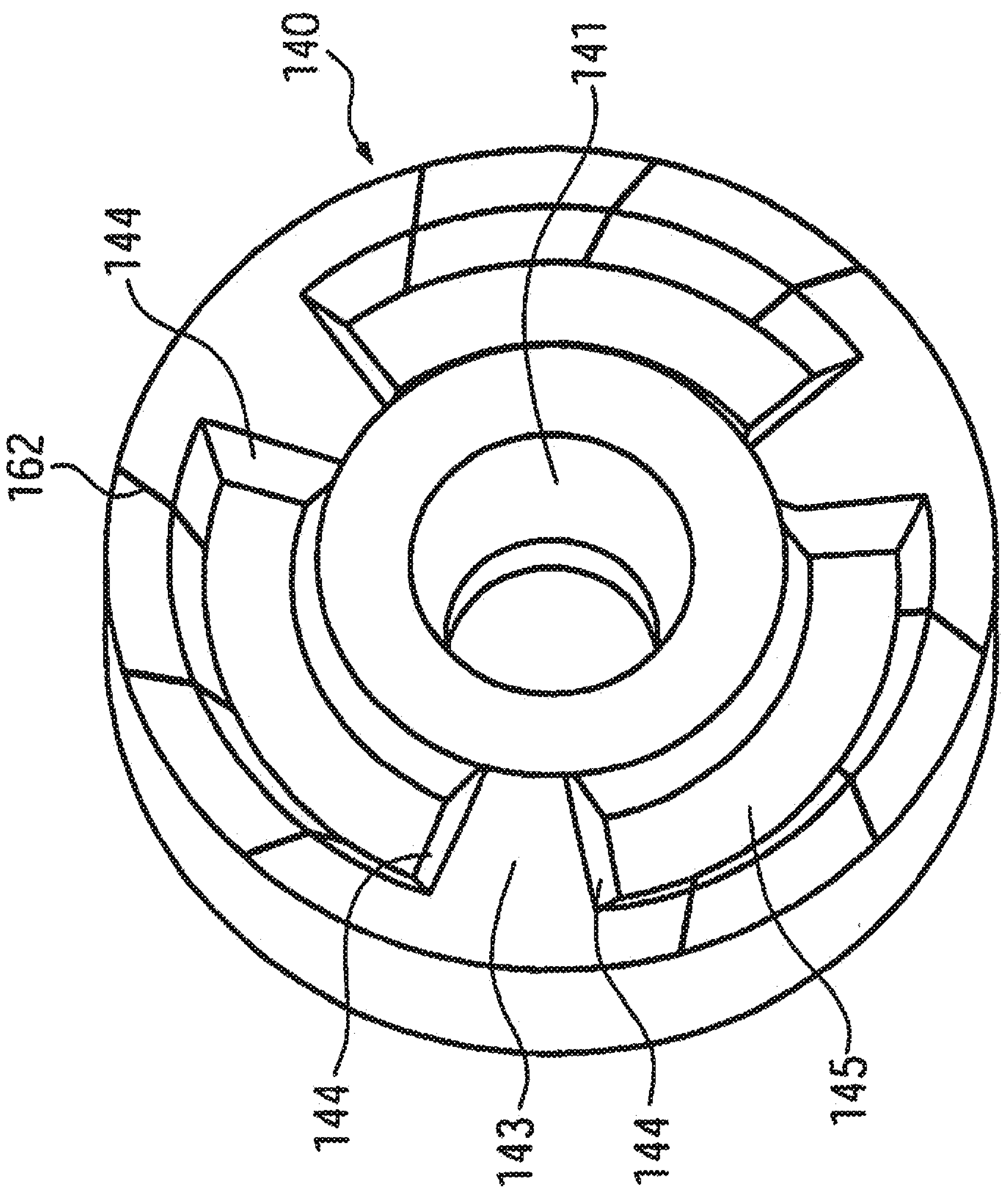

Fig. 9

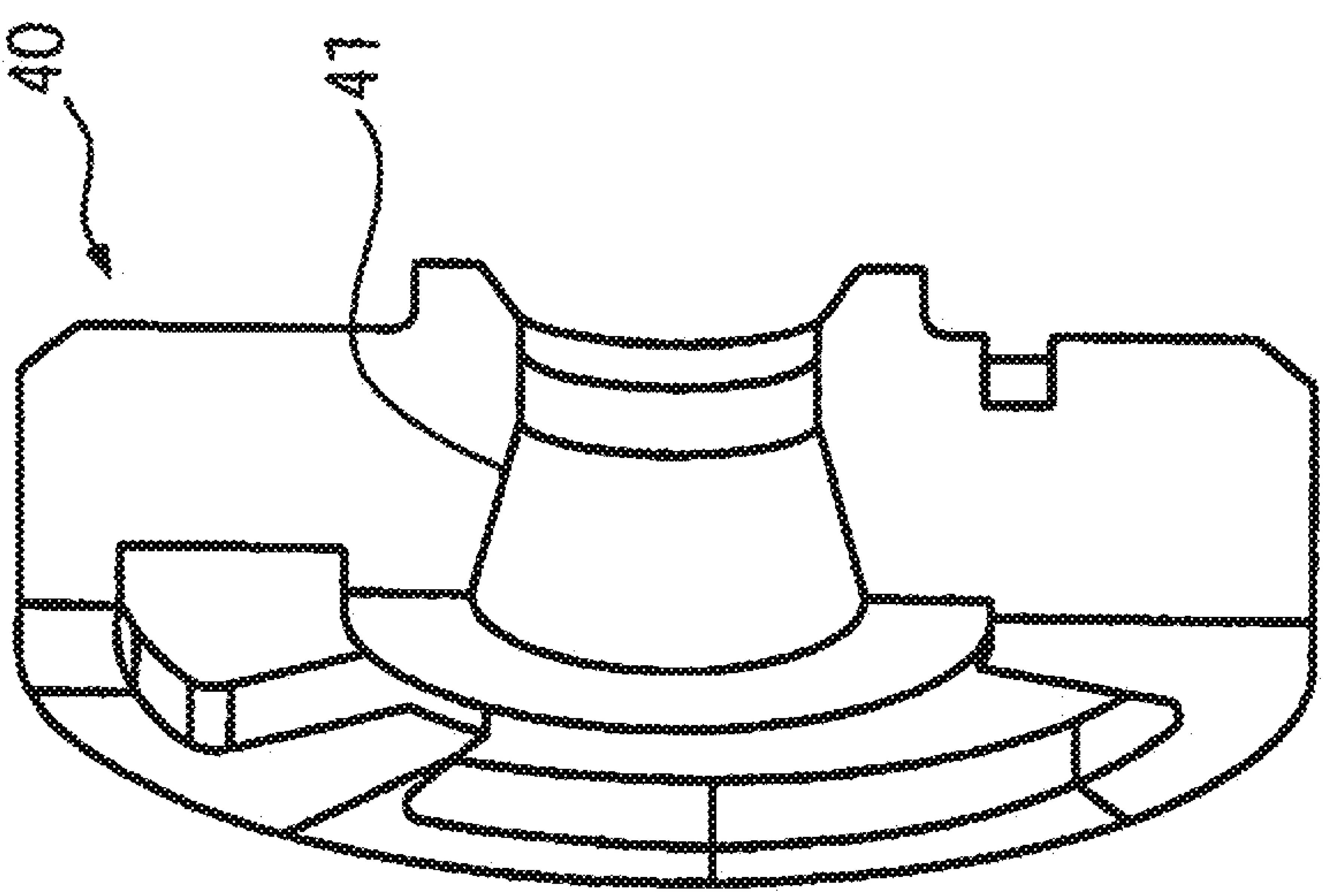

Fig. 10

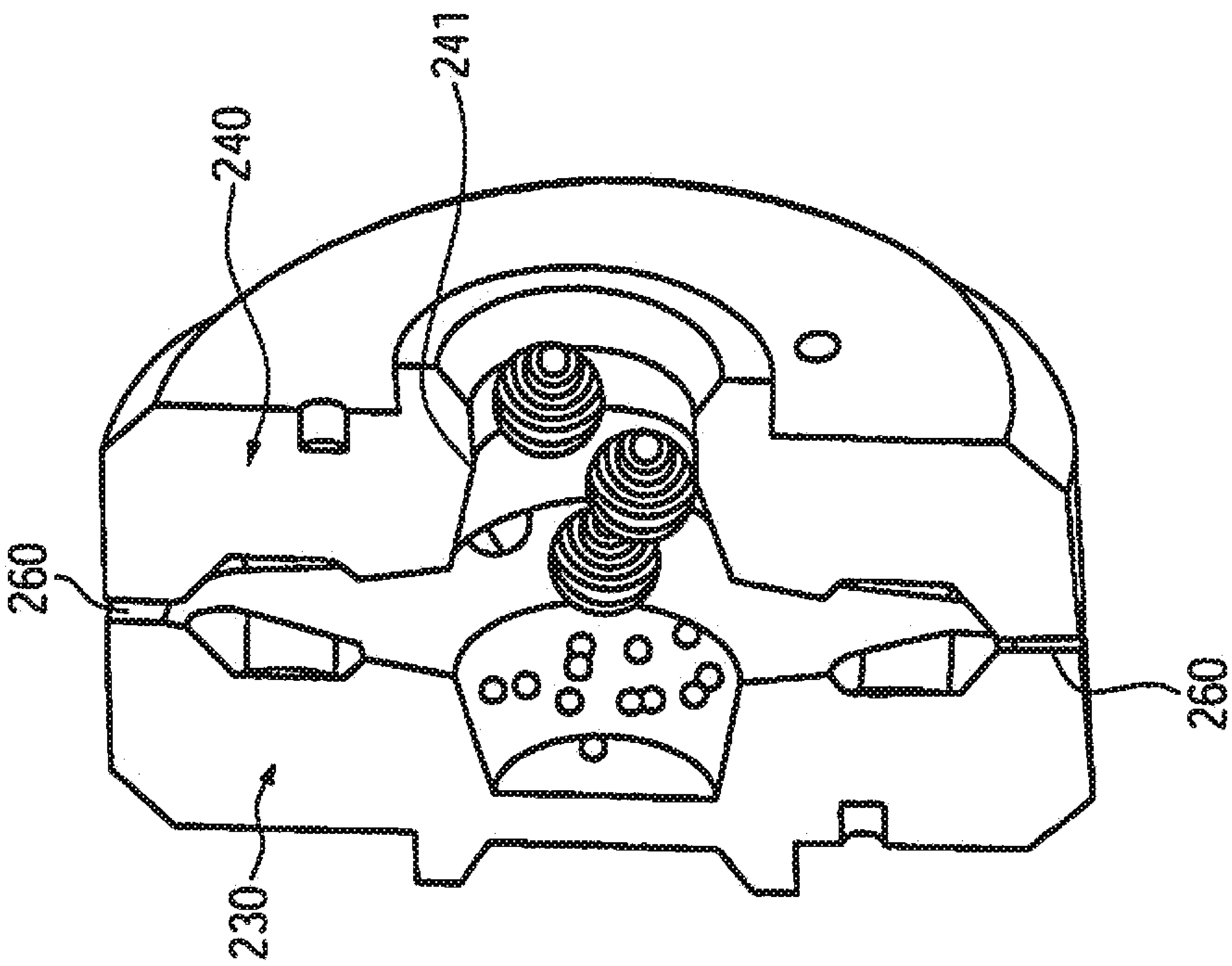

Fig. 11

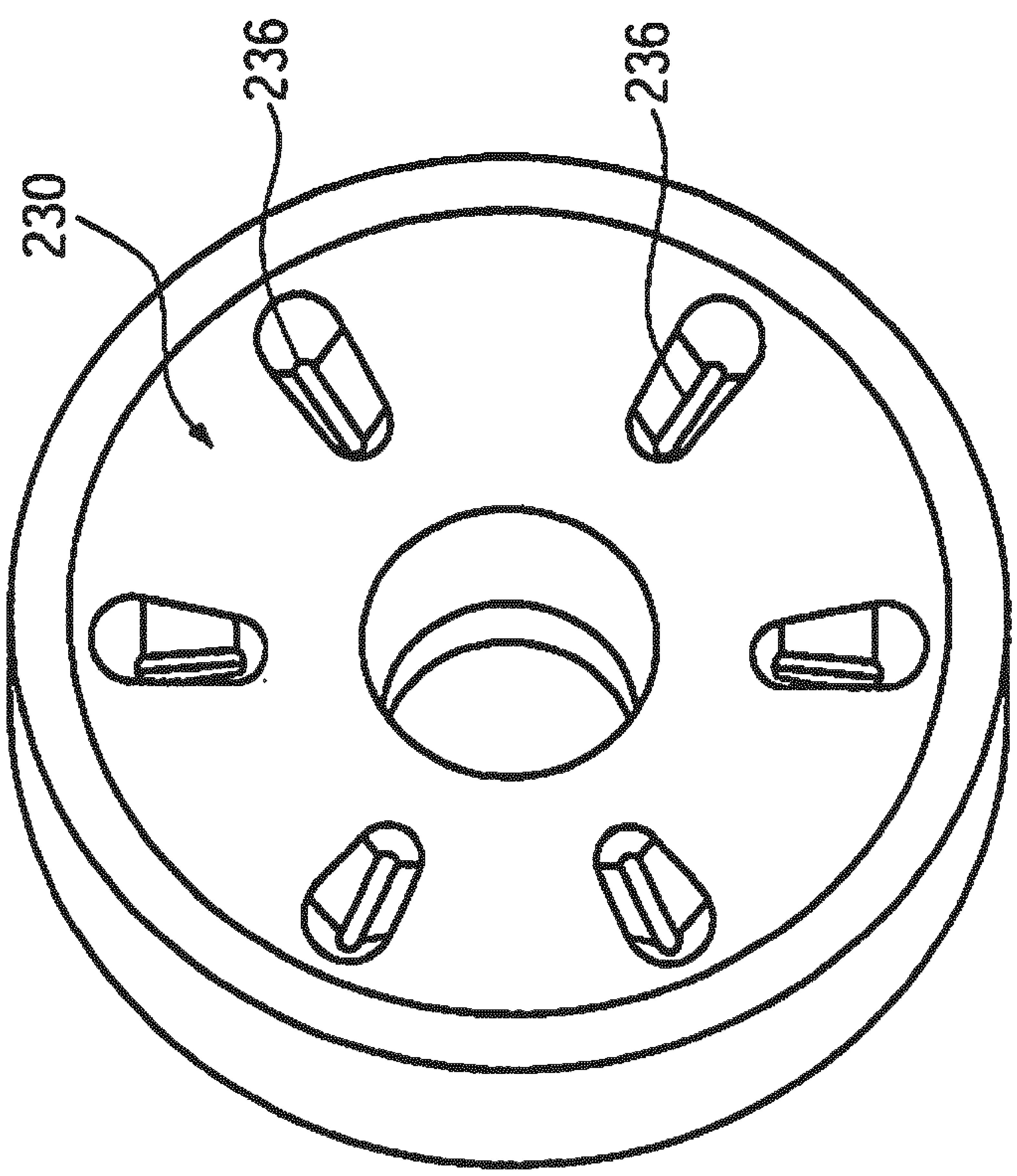

Fig. 12

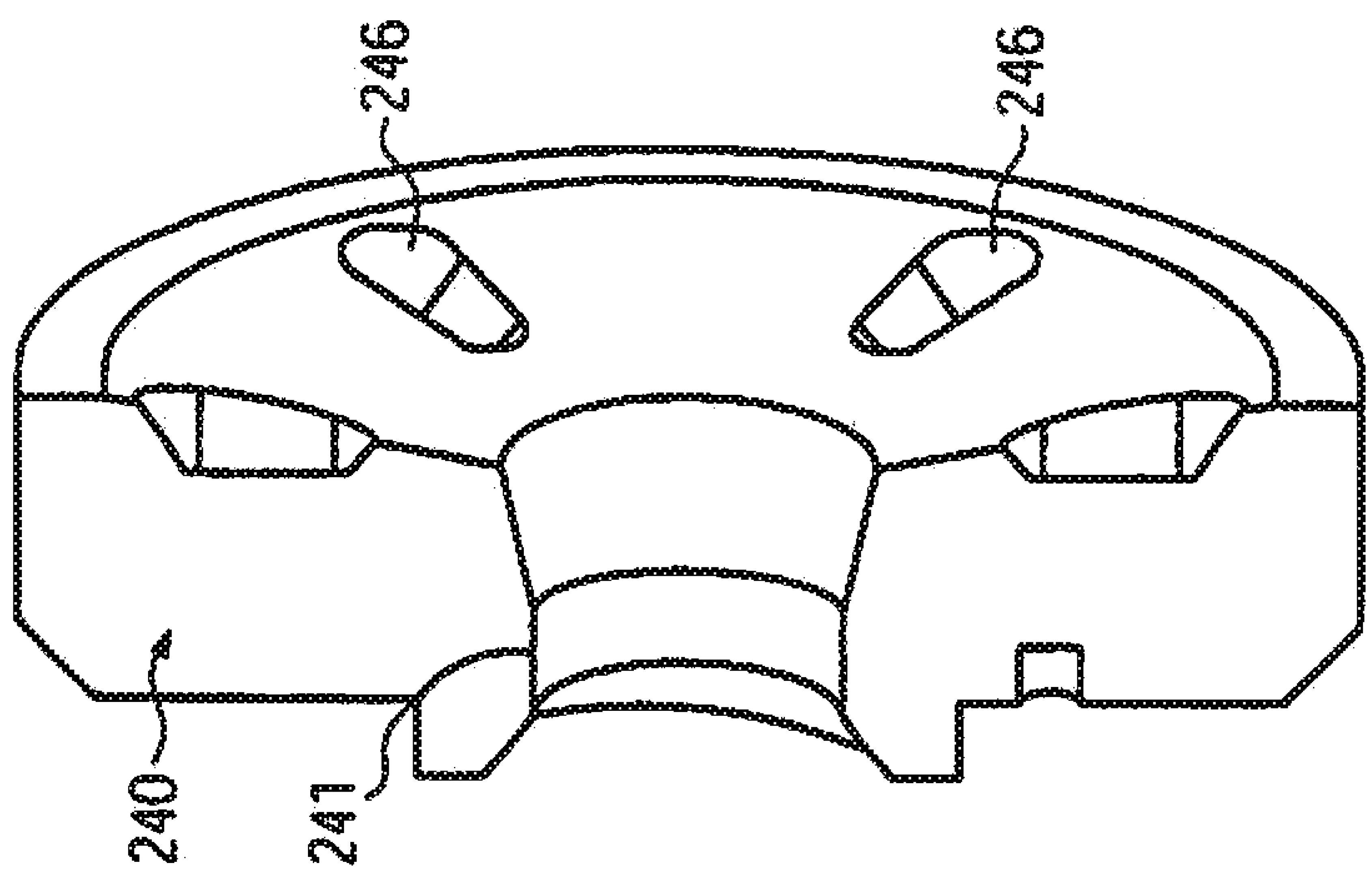

Fig. 13

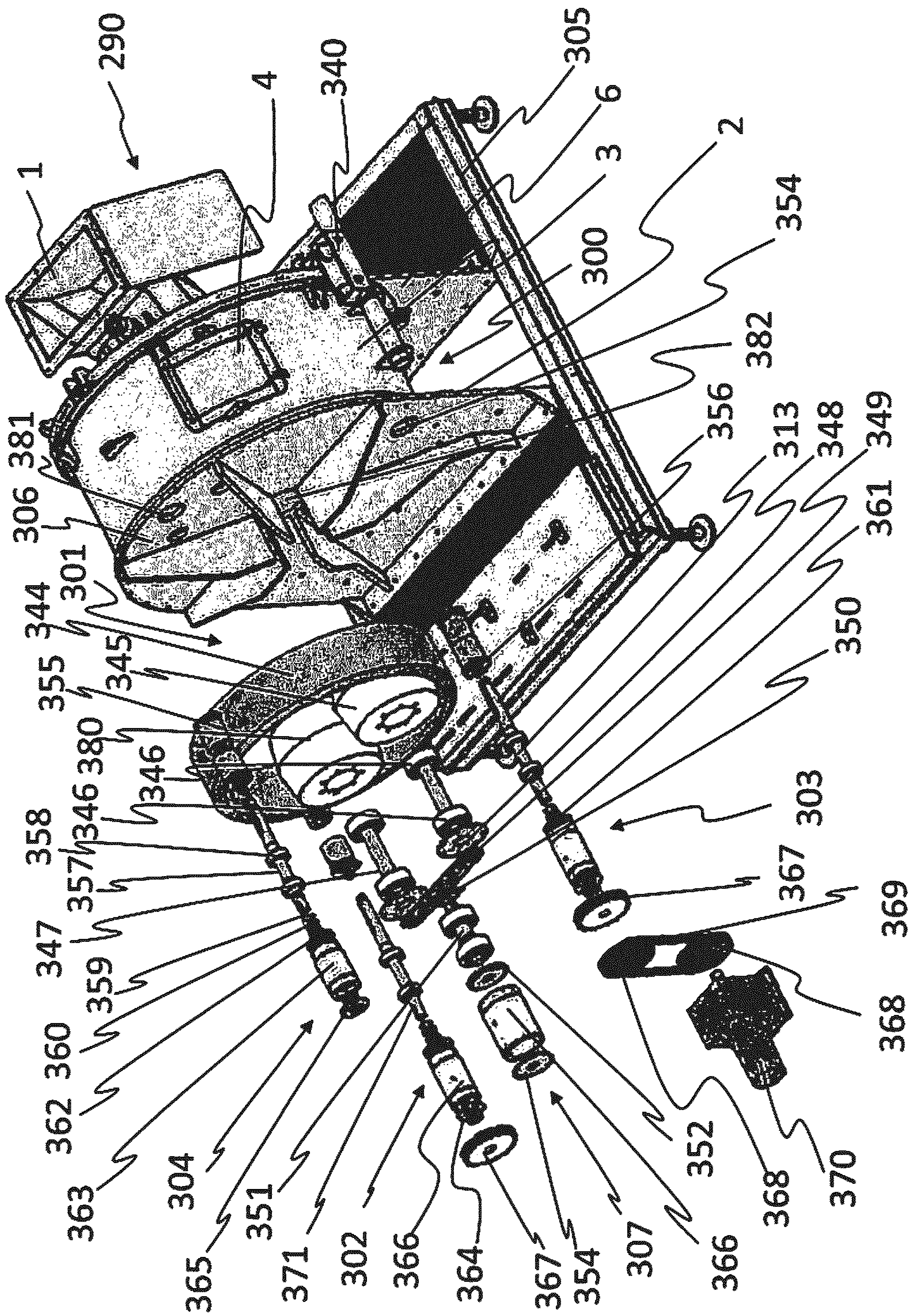
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Fig. 14

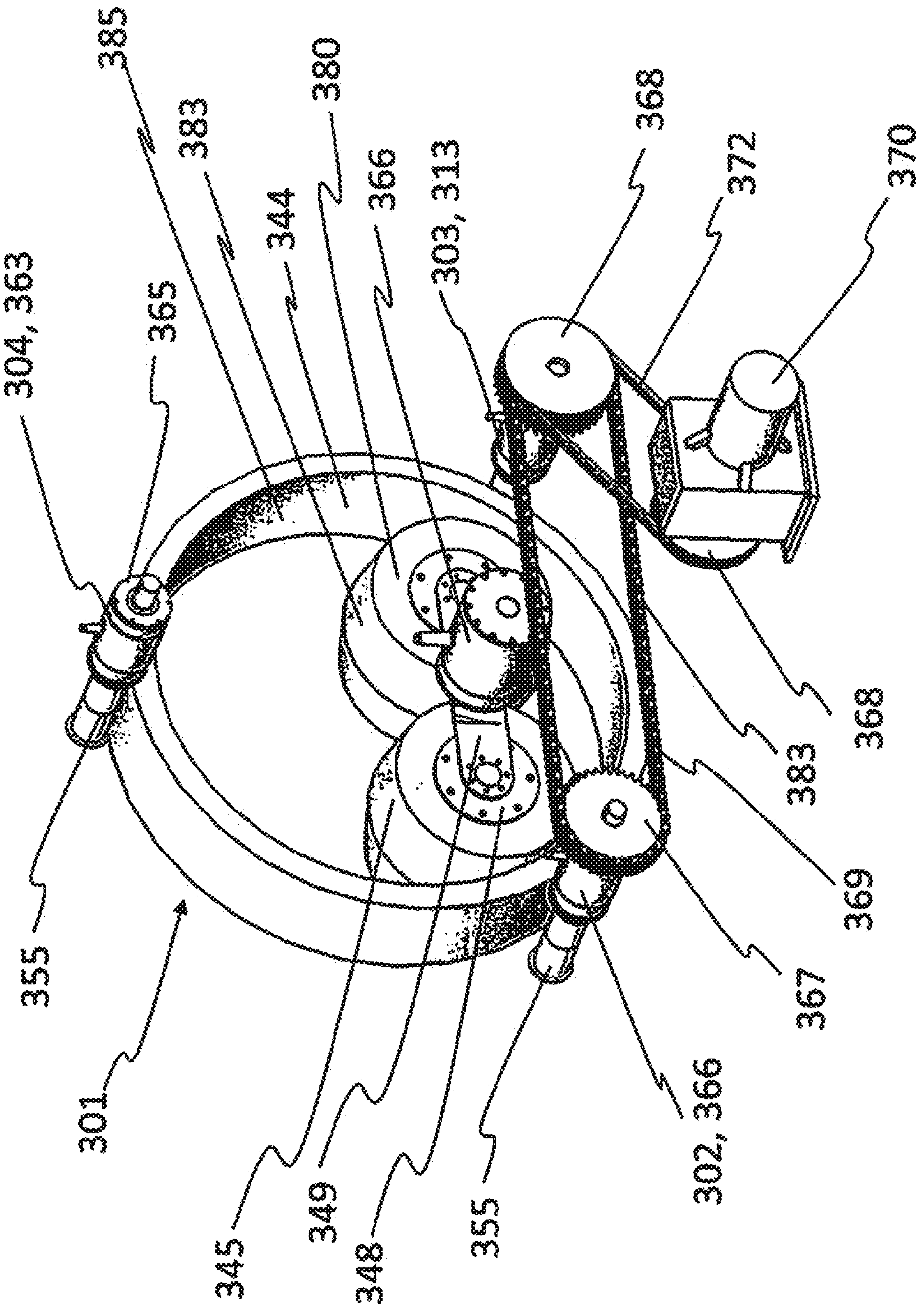
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Fig. 15

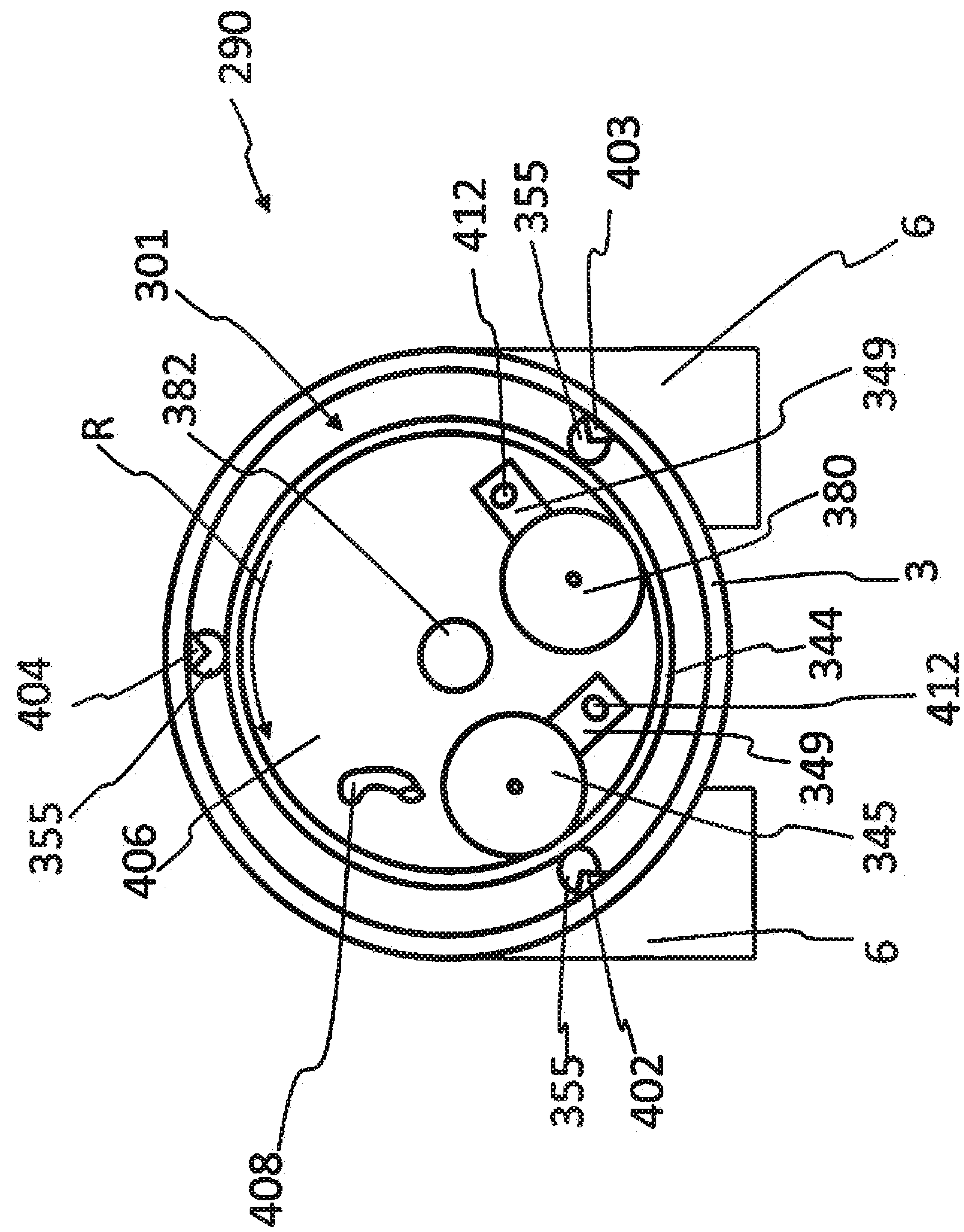
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Fig. 16

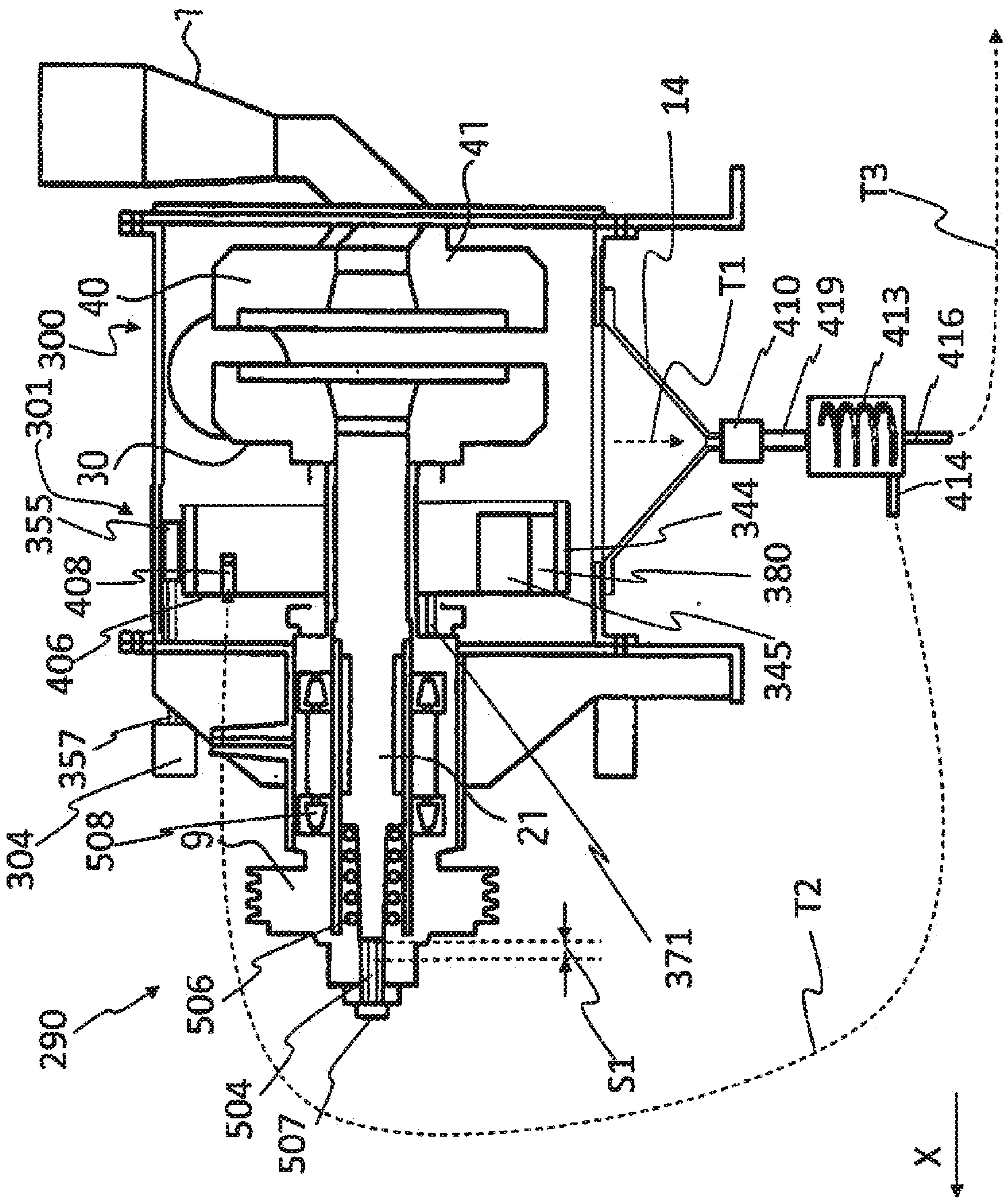
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Fig. 17

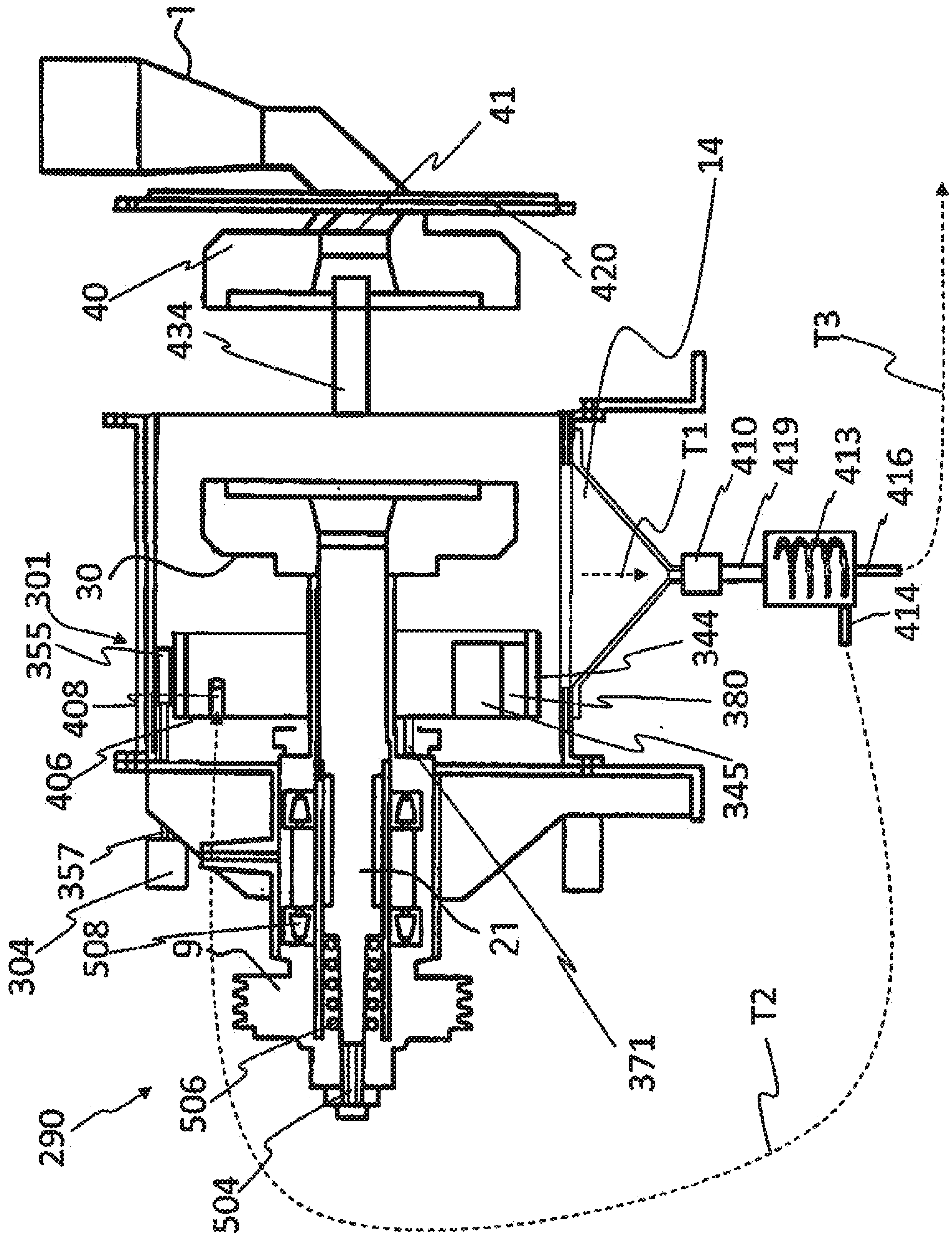
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Fig. 18

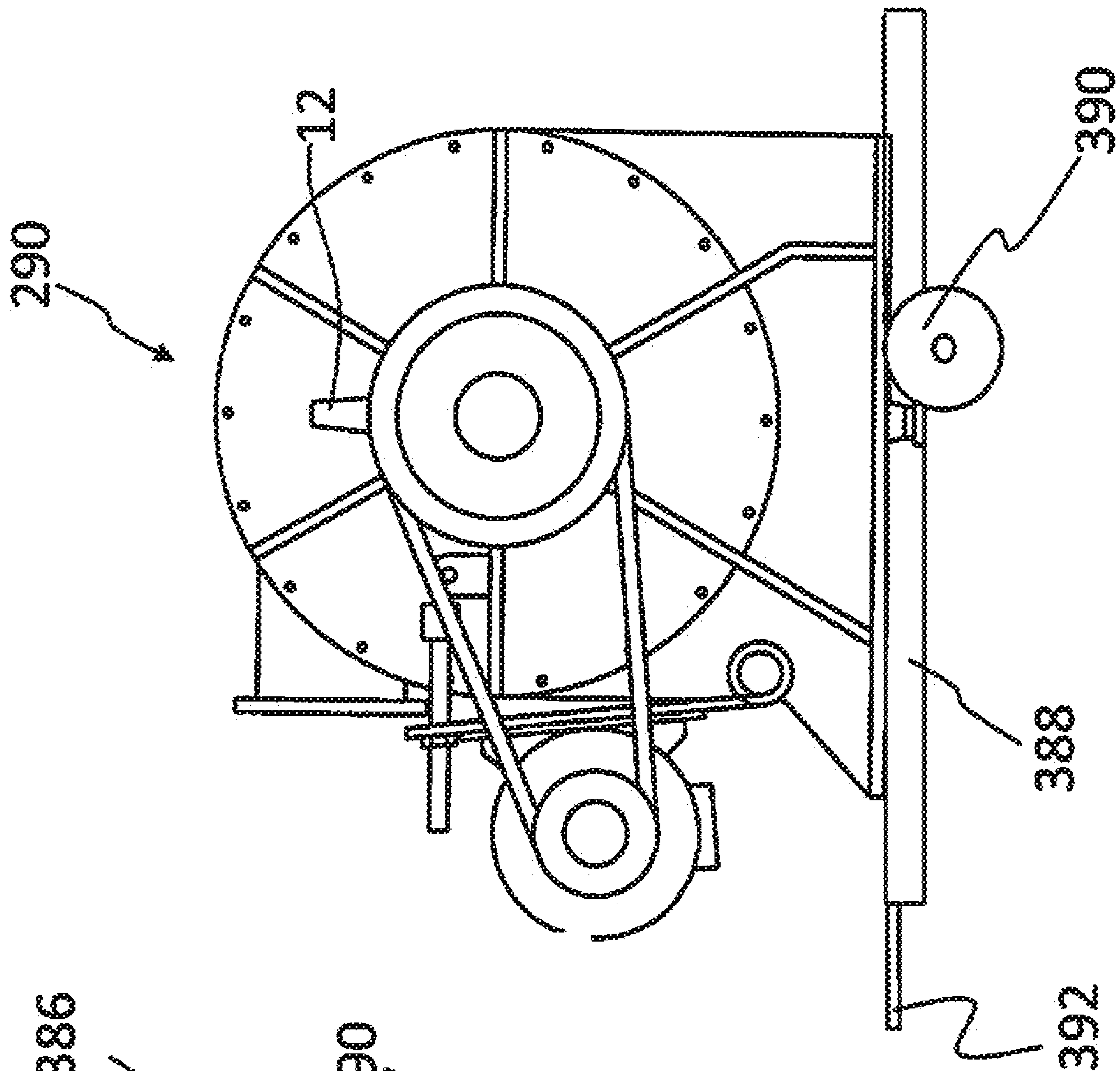

Fig. 19b

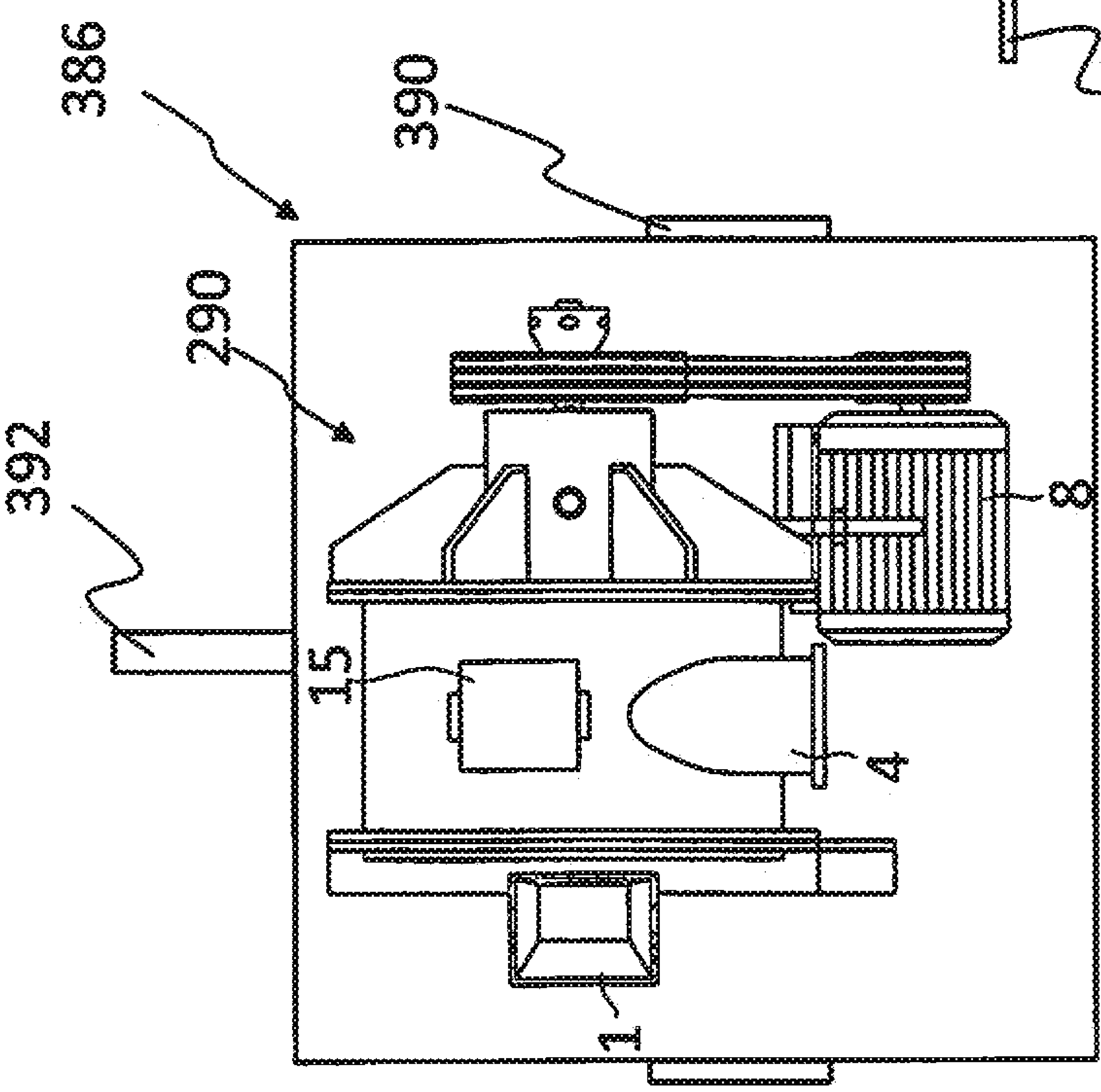

Fig. 19a

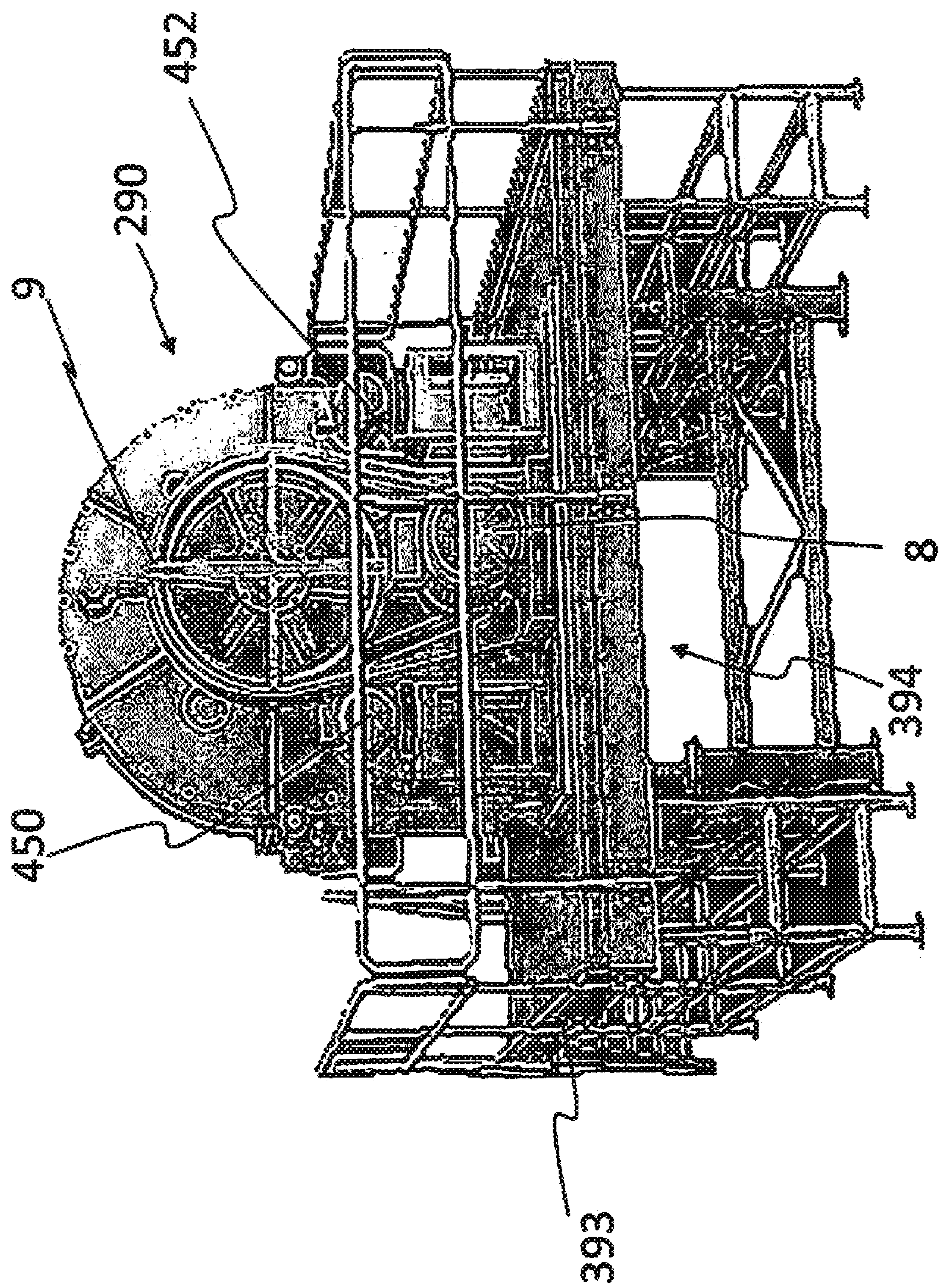

Fig. 20

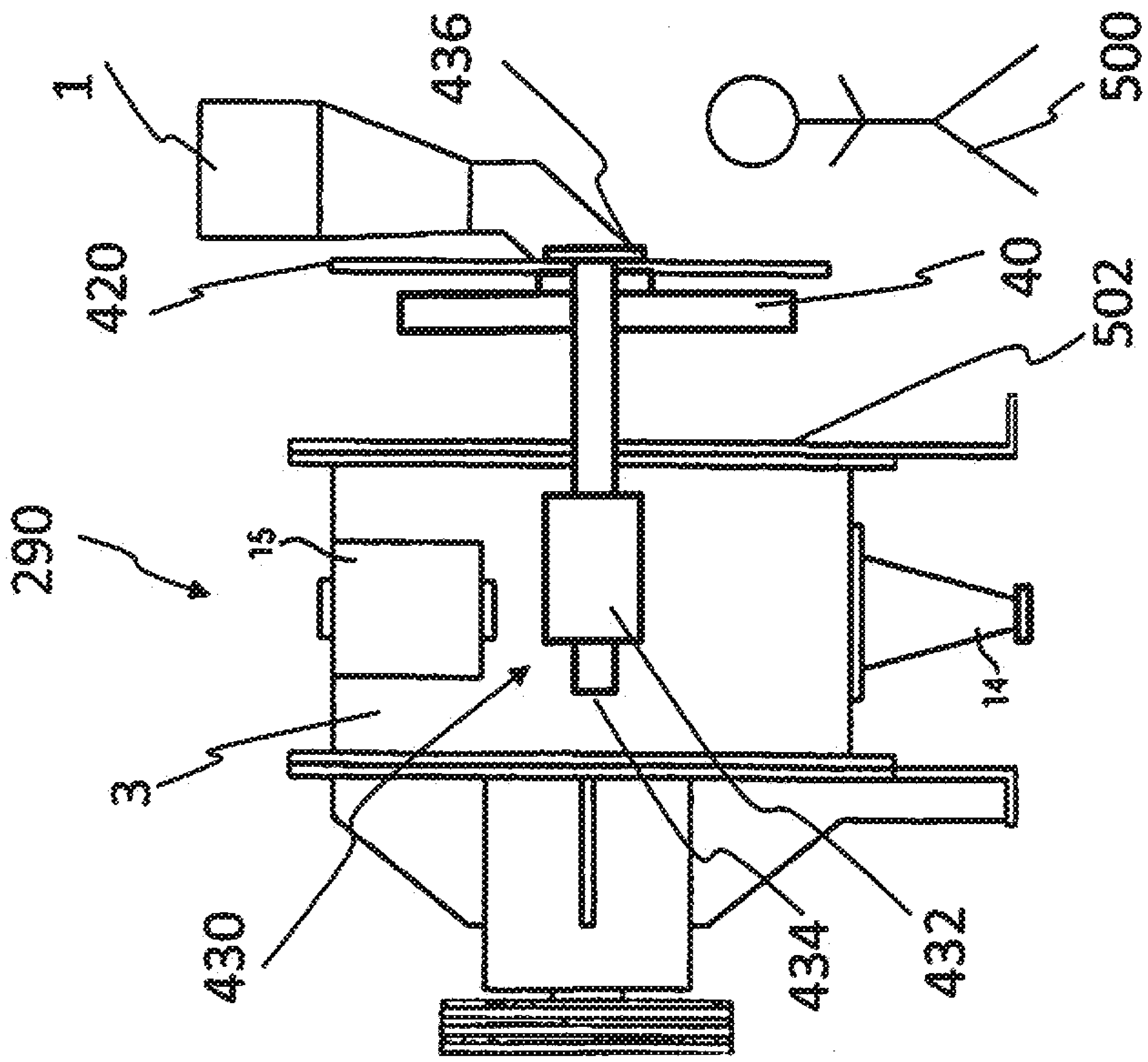

Fig. 21b

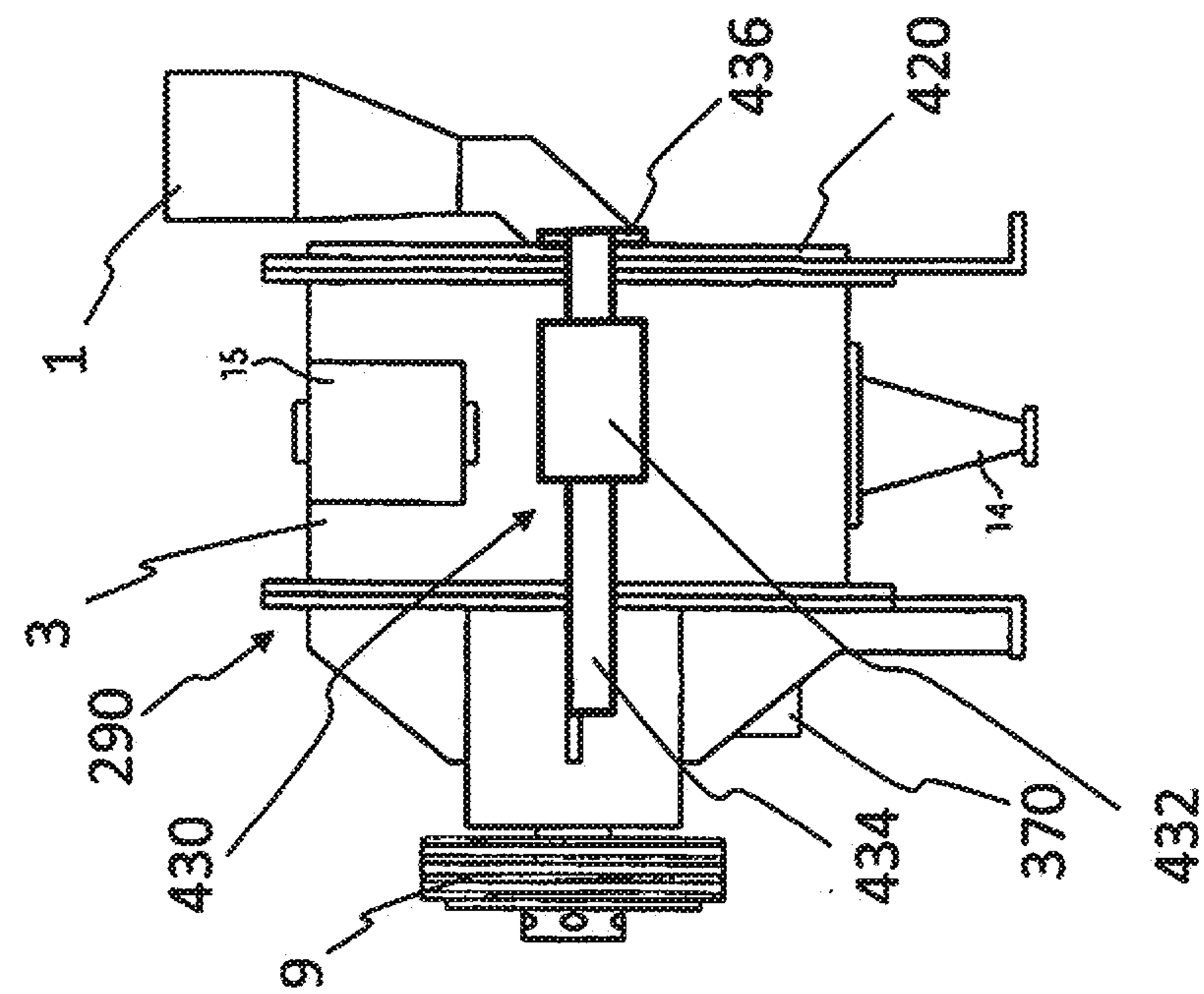

Fig. 21a

DEVICE AND METHOD FOR ORE-CRUSHING WITH A SPRING DEVICE

TECHNICAL DOMAIN

The present invention relates to a method and to a device for comminuting ore or stone and/or slag, the ore being pulverised using water in a wet process or also without using water in a dry process in a particularly ecological manner.

According to the Fraunhofer Institute humanity will consume annually in the year 2050 140 billions tons of minerals, mineral ores, fossil fuels and biomass. Today we consume one third thereof. Resources will become the key in global competition, in particular in mining. "Reducing energie and resources" is deemed to be the maxim of the indsutrie. Energie efficient innovations are a step towards conserving resources and at the same time a chance to change economy and to set sustainable impulses.

Mining plays a strategic role in terms of production of raw materials. Procedural improvements are the first step for more resource usage instead of resource consumption.

Thus, there is a great need to also use environmentally friendly methods and devices when extracting raw materials, in particular in order to protect the people involved from damage to their health. With the conventional comminution of ore the people involved in the mining have their health compromised by the development of dust which may affect the lungs of the people in question.

Furthermore, there is a need to improve the methods and devices used for mining, in particular for the processing of ore, in such a manner that energy consumption is reduced and damage to the environment is minimised.

PRIOR ART

In a classic view dressing of ore takes place until today in four steps. Multiple crushing machines serially connected crush the produced ore to a defined particle size, which is further crushed in mills, mostly ball mills, by wet-mechanical process. The resulting pumpable suspension becomes classified respectively divided in different grain classes. The last step of processing ore rocks forms floating, a physical-chemical process in which ore containing metal is transported in water by means of gas bubbles sticking thereon to the water surface and which are skimmed there. As end product the ore concentrate results.

Those big crushing machines form the preliminary stage of ore dressing in mining. Dependent on country, region, productivity and size of the mine several try working crusher units and a downstream ball mill including the conveyor mechanism and a sieving mechanism form a chain in ore crushing. Size of the facility, energy and logistic effort for the stoneware as well as dust exposure of the environment are enormous in conventional appliances.

The crushing principle of e.g. a jaw crusher only works with mechanically generated pressure. Crushing of crush items mainly happens in a wedge-shaped shaft between a stationary and an eccentric moved crusher jaw. In the course of movement stoneware is crushed until the material is smaller as an adjusted crush gap.

Moreover it continues in a ball mill: In ball mills the precrushed ore rocks are milled together with iron balls in a drum, which is rotated. Thereby the grist is "squashed" by means of the balls, which results in particle crushing. Inclusive an abrasion of the mill balls itself, which contaminate the ore with the iron of the iron balls.

Ball mills for comminuting ore have been known for a long time, the ore being set in rotation together with iron balls until the desired fineness has been achieved in the ball mill. This type of known ball mill is already known from DE 40 02 29, the grinding cylinder containing balls, flints or similar in order to grind up the ore.

However, in such known ball mills the grinding cylinder must be designed to be particularly robust in order to be able to withstand the balls striking against the cylinder wall without any damage, and for this reason the weight of the grinding cylinder is greatly increased. Consequently, the operating costs and energy input are high with such ball mills. Furthermore, the rotating grinding cylinder is subject to a high degree of wear as a result of the balls striking against the grinding cylinder, and so after a relatively short time both the balls and the grinding cylinder have to be replaced. The iron balls cost between 800 US $/ton, depending on the size and property and are in a minimum of time used due to abrasion, wherein the abrasion causes a contamination of the grist and therewith the following floating respectively the floating process is costlier. Moreover, it is necessary with ball mills for the ore to be ground by a separate comminuting unit and then by one or more ball mills connected one behind the other in order to comminute the ore in the desired manner, effective pulverisation of the ore hardly being possible.

Moreover, such ball mills are not suitable for comminuting or pulverising ore together with slag or slag on its own because slag, which is produced in particular as a waste product when further processing ore, is very brittle and has a hard structure.

Further document WO 2011/038914A1 of the same inventor discloses a very good and small size device for comminuting ore. However according to the type of ore, throughput of the device, etc. a danger of overloading the device exists, whereby damage of the same is conceivable.

DESCRIPTION OF THE INVENTION

It is therefore the object of the present invention to provide a method and a device for comminuting ore and/or in particular slag which is highly effective, only shows a small amount of wear and an overloading protection.

This object is achieved by the device according to the features of claim **1** and by the method according to the features of claim **10**.

The invention is based upon the idea of providing a method and a device for comminuting ore, the device according to the invention comprising an ore feed unit for feeding ore to be comminuted to a first comminuting means. The first comminuting means is composed of at least two comminuting elements that can be moved relative to each other, which elements form at least one comminuting space for the ore to be comminuted with each other such that, by a relative movement in the form of a rotation around a rotation axis of at least one of the two comminuting elements the ore to be comminuted is pulverised at east partially in that one or more accelerating elements, in particular protrusions, are provided on at least one of the comminuting elements, said accelerating elements being arranged in particular on the end face of one of the two comminuting elements and accelerating and comminuting the ore to be comminuted by the rotation of one of the two comminuting elements, and wherein between the two comminuting elements and/or in at least one of the two comminuting elements an intermediate space is provided, through which comminuted ore is conveyed during rotation from the center of rotation outwardly and away from the two comminuting elements.

According to the invention at least one of the two comminuting elements comprises a functional connection with a spring means, wherein the spring means is formed in such a way, that it mounts the comminuting element being in functional connection with variably in the direction of the other comminuting element.

This solution is beneficial, since due to the variable mounting of the comminuting element the comminuting element is slideable. Hence, during appliance of forces, which occur during comminuting of ore and which can cause an overloading of the device, the comminuting element is slideable, in particular automatically slideable, whereby immediately unloading of the system respectively the device is caused respectively the occurring forces are reduced.

During a comminuting of ore in the first comminuting means initially a pressure application takes place onto the ore clumps yet slightly or not comminuted. The pressure application is caused by means of a ramp region, which is designed spirally and formed on one or both comminuting elements. Due to the spirally-shaped form a feeding effect is generated during a rotation of a comminuting element, by means of which the ore arranged between the comminuting elements, in particular between the ramp region of one comminuting element and a correspondent region of the other comminuting element, is compressed respectively applied with increasing pressure. The pressure applied to the ore clump normally causes that the ore clumps fall into pieces and thus yield to the pressure. In case of presence of ore clumps not falling into pieces, the generated pressure threatens to further increase, whereby the load of the device components, in particular of the comminuting elements, the actuating shaft, the bearings, etc. also rises significantly and can even reach a level above which damage of individual or several of said components is possible. Due to the usage of a spring means according to the invention overloading of the components during operation of the first comminuting means can be avoided. Because, the spring means deflects in case the load becomes too high respectively exceeds a defined, in particular adjusted, level. A sliding of one comminuting element results due to the deflection of the spring means, whereby the comminuting elements are spaced apart from each other. After respectively in case of a pressure drop between the comminuting elements the deflected spring means causes a returning of the comminuting element into the initial position. Due to the sliding of the comminuting element a slit between the comminuting elements was enlarged, through which larger ore particles respectively ore clumps can get out of the first comminuting means. All ore particles respectively ore clumps which got out of the first comminuting means are guided to a separating means, due to which separation of the already sufficiently comminuted particles and the not yet sufficiently comminuted particles respectively ore clumps takes place. The not yet sufficiently comminuted ore particles respectively ore clumps are then again fed to the first comminuting means or to a second comminuting means.

Further, it is also conceivable that ore particles respectively ore clumps can be present in the region of the comminuting protrusions of the comminuting elements and do not fall into pieces due to the applied pressure. Since the comminuting protrusions of the comminuting elements are arranged radially spaced apart from the center of the comminuting protrusions the ore particles respectively ore clumps are causing high momentums in this region, which can lead to a damaging of the first comminuting means, in particular of one or both comminuting elements, the actuating shaft, etc. The arrangement of a spring means according to the invention enables, preferably also in case, that a deflection of a comminuting element takes place, in particular the comminuting element which is coupled with the shaft.

Further beneficial embodiments of the inventive device and the inventive method result from the dependent claims and/or from the following specification.

According to a preferred embodiment of the present invention at least one of said comminuting elements is arranged at the shaft for actuating the comminuting element, wherein the spring means is directly coupled with the shaft or the comminuting element and is pretensioned by it and wherein the shaft and the comminuting element arranged thereon are slideable opposite to the spring force of the spring means. This embodiment is beneficial, since a protection of the comminuting elements and the shaft, which is connected with one comminuting element, is in particular caused thereby.

A sliding of the shaft and the comminuting element takes place according to a further preferred embodiment in dependency of the pretension of the spring means, wherein a deflection of the spring means results during operation of the first comminuting means because of a deflection force generated between the two comminuting elements and directed opposite to the contact pressing force resulting from the spring force, in case the deflection force exceeds the contact pressing force. This embodiment is beneficial, since the spring force preferably serves as essential parameter for position changes of the shaft and/or the comminuting element. The spring force is preferably arbitrarily modifiable, whereby optimized adjustments respectively configurations are foreseeable for most different operation and/or boundary conditions.

According to a further preferred embodiment of the present invention the spring means comprises a mechanical spring means, in particular a spiral spring, a pneumatic spring means and/or a hydraulic spring means. This embodiment is beneficial, since the spring means is provideable with respect to operation and/or boundary conditions, whereby the device according to the invention is optimal adjustable.

The spring means has multiple suspension means, wherein the single suspension means are arranged in such a manner that they are pushing the comminuting element coupled with the shaft into the direction of the other comminuting element. This embodiment is beneficial since the different suspension means can be shaped equally or differently, whereby a very precise adjustment of the desired overall spring forces is in return causeable.

According to a further preferred embodiment of the present invention the shaft is mounted in a housing of the device by means of roller bearings and coupled with a actuating means for rotating the shaft and the comminuting element arranged thereon. The mounting by means of roller bearings is beneficial since roller bearings can handle high forces and are very good adjustable. Furthermore, this embodiment is beneficial since the roller bearings are preferably arranged in the housing of the device according to the present invention and are thus protected against environmental influences.

The spring means is arranged in an end region of the shaft respectively coupled with the shaft according to a further preferred embodiment of the present invention, wherein the end region is spaced apart from a second end region of the shaft, on which the comminuting element is arranged. Between the end regions of the shaft are preferably arranged the roller bearing for mounting the shaft. Further, preferably in the region of the end in which the spring means is provided also an actuating means respectively a coupling with a actuating means is provided. This embodiment is beneficial since the spring means is preferably as much as possible spaced apart from the comminuting element to preferably avoid damaging or negative functional impacts due to the comminuted ore.

According to a further preferred embodiment of the present invention the comminuting element
is arranged in the direction of extention of the rotational axis at a housing of the device at least time-wise closing housing cover, wherein the housing cover is moveable with respect to the device and wherein the fixed arranged comminuting element is pressed against the other comminuting element by means of the spring means, which connects the housing cover with the device. The spring means is preferably formed as hydraulic spring means and is particular preferably formed by a hydraulic means, which also enables a displacement of the housing cover for opening and closing of the housing for e.g. maintenance work. It is also conceivable that the comminuting element arranged at the housing cover is mounted respectively pretensioned via a spring means and the comminuting element arranged at the shaft is mounted respectively pretension by means of a further spring means.

The spring constant of the spring means, the sliding path of the comminuting element and/or the deflection path of the spring means are changeable, in particular adjustable or exchangeable, due to a further preferred embodiment. Adjustable means hereby e.g. that due to a manipulation of the present items a change of the further variables takes place. Thus, e.g. in case a mechanical spring is provided it is e.g. manipulatable respectively compressible by means of a screw, whereby the potential deflection path decreases. Further in case of presence of e.g. a pneumatic spring the pressure in a pneumatic cylinder is changeable. A change of one of the mentioned variables by means of exchanging a component means the replacing of said component by another component with preferably other physical and/or mechanical properties. So, e.g. in presence of a mechanical spring another mechanical spring is usable, which consists of another material, is larger, has another form, etc.

Furthermore, it is conceivable that the sliding path of the comminuting element being in a functional connection with the spring means is during operation of the first comminuting means less than 5 cm and preferably less than 3.5 cm and particular preferably less than 1 cm. Further it is conceivable that the contact pressing force generated by the spring means amounts at least 1000 N, preferably at least 2000 N and particular preferably at least 10000 N.

Furthermore, the subject-matter of a further patent application filed by the same applicant at the same day by the same patent office, which also refers to a device and a method for ore comminuting is fully incorporated into the subject-matter of the present patent application by reference.

Individual or all representations of figures described in the following are preferably considered as constructional drawings, that means that the dimensions, proportions, functional contexts and/or arrangements correspond preferably exactly or preferably essentially to those of the device according to the invention respectively the products according to the invention.

Further benefits, goals and features of the present invention will be described by the following specification of the attached figures, in which exemplarily devices for crushing ore according to the invention are illustrated. Components of the device according to the inventions, which match at least essentially with respect to their function can be marked with the same reference sign, wherein such components do not have to be marked or described in all figures.

In the following the invention is just exemplarily described with respect to the attached figures.

In the following the invention will be described, purely by way of an example, by means of the attached figures.

FIG. **1** shows a perspective view of a part of the device according to the invention;

FIG. **2** shows an exploded representation of a part of the device according to the invention of FIG. **1**;

FIG. **3** shows a top view of a part of the device according to the invention of FIG. **1**;

FIG. **4** shows a side view of a part of the device according to the invention of FIG. **1**;

FIG. **5** shows a part of the side view of FIG. **1**;

FIG. **6***a* shows a part of the device according to the invention of FIG. **1**, partially as a cross-section;

FIG. **6***b* shows the illustration of FIG. **6***a* broadened by a separator and respective components

FIG. **7** shows diagrammatically the two comminuting elements of FIG. **6** as a cross-section;

FIG. **8** shows the two comminuting elements of FIG. **7** in an opened up position;

FIG. **9** shows a comminuting element analog to FIG. **8**, illustrated diagrammatically;

FIG. **10** shows the comminuting element of FIG. **8**, partially as a cross-section;

FIG. **11** shows further embodiments of the comminuting elements for the part of the device according to the invention shown in FIG. **6***a;*

FIG. **12** shows diagrammatically a comminuting element of FIG. **11**; and

FIG. **13** shows the other comminuting element of FIG. **1**, partially as a cross-section.

FIG. **14** shows a perspective view of the inventive device in an exploded view;

FIG. **15** shows a perspective view of a preferred embodiment of a second comminuting means of the device according to the invention;

FIG. **16** shows a schematic view of the second comminuting means;

FIG. **17** shows a schematic cross-sectional view of the ore comminuting device according to the invention;

FIG. **18** shows the illustration of FIG. **17** in an opened configuration;

FIG. **19***a* shows a schematic illustration of a device according to the invention on a transportation means in a top view;

FIG. **19***b* shows a schematic illustration of a device according to the invention on a transportation means in a side view;

FIG. **20** shows a device according to the invention on a platform;

FIG. **21***a* shows a device according to the invention in a closed configuration and with a closing means; and

FIG. **21***b* shows a device according to the present invention in an opened configuration.

DESCRIPTION OF A PREFERRED EMBODIMENT

According to FIG. **1** the device according to the invention is illustrated, the ore to be comminuted respectively the slag to be comminuted being introduced into a funnel or feed funnel **1** which constitutes the ore feed unit. Alternatively, instead of a funnel a screw conveyor can also be provided which feeds the ore to be comminuted under pressure into the first comminuting means. The ore is fed through the funnel **1** to the cylinder-like housing **3** which is mounted on one foot **2** and one foot **6**. The pulverisation of the ore to be comminuted takes place in this housing **3**. Here a motor **8** transfers the torsional moment from the motor **8** to the pulveriser by means of a drive roller **11** and a belt **10** and a belt pulley **9**.

As can be gathered in particular from FIG. **2**, a suction opening **4** is optionally possible through which the pulverised ore can be sucked out by means of negative pressure. Alternatively, and in particular as a rule, there is provided in the lower region of the housing **3** an outlet funnel **14** which generally forms the first outlet unit. By means of this outlet funnel **14** the pulverised ore is discharged from the device according to the invention with the aid of the force of gravity or by suction.

A control flap **15** can be provided on the housing **3** in order to provide, if so required, access to the interior of the housing. However, this is not necessary for the function of the device according to the invention. As can be gathered in particular from FIG. **3**, the control flap **15**, like the feed funnel **1**, is disposed in the upper region of the device according to the invention. Furthermore, the ore can be fed in a continuously manner to the first comminuting means through the feed funnel or also in a non-continuously manner to the first comminuting means if ore or slag is only fed sporadically to the device according to the invention.

FIGS. **4** and **5** respectively show a side view of the device according to the invention from which it is evident that the outlet funnel **14** is provided in the lower region of the cylinder-shaped housing **3**.

One can see in particular from FIG. **6***a* the function and the structure of the pulveriser. The belt pulley **9** is, as already described, driven by the motor **8** and transfers this torsional moment via a shaft **21** onto a comminuting element **30** which is thus rotating. In its simplest form the comminuting element **30** is designed as a rotating turning element **30** with a disc-like configuration which together with a stationary fixed element **40** forms the first comminuting means **300**. As can be seen from FIG. **6** the ore to be comminuted is fed via the inlet funnel **1** into the housing **3** by a feed opening **41** being provided substantially in the centre of the fixed element. The ore fed through the feed opening **41** is now pulverised between the fixed element **40** and the rotating turning element **30** and expelled or conveyed away radially outwards in pulverised form between the two comminuting elements **30**, **40** and collected within the housing **3** in pulverised form and then discharged from the outlet funnel **14**.

Observing in detail the path of the material respectively rocks in the device according to the invention, thus primarily material respectively the stones get into the devices via a feed funnel. Via outlet opening in the centre of the fixed disc jaw respectively the fixed comminuting element **40** material enters the intermediate space, wherein the actuated disc jaw respectively the comminuting element **30** causes the acceleration of material respectively stoneware. Into the geometry of the disc jaws **30**, **40** carrier elements are preferably integrated, which transfer the carried ore stones in a radial speed. With the gathered acceleration energy are the stones colliding with each other and that causes highly efficient comminuting of mill material.

This Micro Impact is based on accelerated material by means of a relative movement of the comminuting elements **30**, **40** respectively the jaws and due to the narrowness of the intermediate space comminuting takes place in very fast time intervals. The carrying elements on the disc jaws **30**, **40** ensure high speeds in radial direction as well as in axial direction, thus that as a result the generated powder is pressed outwards of the intermediate space and gets as powder via outlet funnel **14** for further processing out of the device **290**. The degree of comminution—respectively the grain size—in particular defines the distance of both disc jaws respectively of both comminution elements **30**, **40**. The smaller the distance the finer the grain size. The work process further decreases by adding water into the mill. Therefore, the operating staff has multiple parameters for adjustment for the required grain size—and this without any dust exposure.

The device according to the invention of FIG. **6***a* is illustrated modified in FIG. **6***b*. According to this illustration a pumping means **410** is connected to the outlet funnel **14**, in turn a separating means **413** is connected to the pumping means **410**. The ore feeded via outlet funnel **14** to pumping means **410** is preferably accelerated and/or pressure is applied to it by means of pumping means **410** and via conduit section **419**, in particular a pipe or a hose, feeded into the separating means **413**. It is also conceivable, that pumping means **410** is directly respectively straight connected with separating means **413**. Ore is outputted via the first outlet **414**, which again shall be fed to the first comminuting means, in particular the comminuting elements **30**, **40**. The feeding of the ore outputted via the first outlet **414** happens preferably transport path T**2**, that means the ore to be further comminuted is preferably fed to feeding funnel **1**. Housing **3** particular preferably comprises the first comminuting means **300** and/or the feeding funnel **1** a feeding connection **520** via which flowable substances are feedable to the first comminuting means **300**. In particular ore fed via T**2** is hereby considered as flowable substance. Further, feeding connection **520** can comprise multiple connection spots for coupling one or a plurality of further conducting elements. Hence, it is also conceivable that a conduit respectively a conduit element for feeding a liquid, in particular water or a water comprising liquid, is coupled via feeding connection **520** with the device **290** according to the invention. The separating means **41** preferably has a second outlet **416** from which already sufficiently comminuted ore is outputted. The sufficiently comminuted ore respectively ore which does not shall or must be fed to the first comminuting means **300**, that means comminuting elements **30**, **40**, preferably gets according to transport path T**3** directly conducted to a further processing means, in particular a second comminuting means (cf. FIG. **17**) or a floating means.

Further, FIGS. **6***a* and **6***b* show a spring means **504** schematically in the area of a first axial end **521** of shaft **21**. The spring means **504** can be formed e.g. as mechanical, pneumatical or hydraulic spring means and is preferably arranged between belt pully **9** and shaft **21**. However, it is conceivable that the spring means **504** can be formed respectively arranged at other positions in the area of shaft **21**. Reference number S1 characterizes a displacement range, on which shaft **21** is moveable respectively between which shaft **21** is variably mounted, in case shaft **21** is moved in axial direction and a deflection of spring means **504** is caused.

During a comminution of ore in the first comminuting means **300** an initial pressure application on the ore clumps yet only a little or not comminuted takes place. The pressure application is caused by a ramp region **31**, which is designed volutely and formed at one or both comminuting elements **30**, **40**. Due to the voluted design a feeding effect is caused by a rotation of a comminuting element **30**, due to which ore between the comminuting elements **30**, **40**, in particular between the ramp region **31** of a comminuting element **30** and a corresponding region **42** of the other comminuting element **40**, is compressed respectively applied to increasing pressure. Pressure applied to ore clumps normally causes that the ore clumps are falling apart in very small pieces and therefore succumb to the pressure. In presence of ore clumps which do not succumb the generated pressure threatens to further increase, whereby the workload on the device components, in particular comminuting elements **30**, **40**, shaft **21**, bearings **506**, **508**, etc. also strongly increases and can even reach a level, from which damage of single or multiple of said components is possible. Due to the inventive utilization of a spring means **504** overloading of the components in the range of the first comminuting means **300** can be prevented. There is to say, the spring means **504** deflects in case the workload is to high respectively surpasses a specific, in particular adjusted, level. Because of the deflection of spring means **504** a sliding of a comminuting element **30** results, whereby the comminuting elements **30**, **40** are spaced apart from each other. After respectively during a pressure decrease between comminuting elements **30**, **40** the deflected spring means **504** causes a return of the comminuting element **30** in the starting position. Due to the sliding of the comminuting element **30** a slit between the comminuting elements **30**, **40** is increased, whereby larger ore particles respectively ore clumps can escape from the first comminuting means **300**. All ore particles respectively ore clumps escaping from the first comminuting means **300** are fed to a separating means **413**, by means of which a separation of the already sufficient comminuted particles and the not yet sufficient comminuted particles respectively ore clumps are caused. The ore particles respectively ore clumps not yet sufficiently comminuted are again fed to the first comminuting means **300** or to a second comminuting means **301**.

Further, it is also conceivable that ore particles respectively ore clumps can occur in the region of comminuting protrusions **35**, **45** and do not fragment in consequence of the applied pressure. Since the comminuting protrusions **35**, **45** of comminuting elements **30**, **40** are radially spaced apart from the centre ore particles respectively ore clumps in this region cause the generation of high momentums, which can cause damaging of the first comminuting means **300**, in particular of one or both comminuting elements **30**, **40**, shaft **21**, etc. The inventive arrangement of a spring means **504** enables preferably also in that case, that a deflection of a comminuting means **30**, **40**, in particular a comminuting element **30**, which is coupled with shaft **21**, takes place.

The inventive manner of comminuting only requires a short time due to the small floor requirements of the comminuting space, wherein the comminuted ore is fed to the outside through the intermediate space **60** between the comminuting elements **30**, **40** during a rotation of the rotation element and away from both comminuting elements **30**, **40**, as it is e.g. illustrated by comminuted ore **55** in FIG. **7**. This means, that ore clumps are comminuted by means of the relative movement in form of a rotation between the two comminuting elements **30**, **40**, wherein according to a further embodiment two comminuting elements **30**, **40** can be used with different rotational speeds as well as equal or opposed directions of rotation.

The pulverisation is described in more detail, in particular with regard to FIG. **7**. In the same way as in FIG. **6***a* the ore to be comminuted is fed via the feed opening **41**, which is preferably located substantially in the centre of the fixed element preferably being formed as comminuting section **40**, into a comminuting space between the fixed element **40** and the turning element **30**. FIG. **7** shows by way of example several lumps of ore **50** which represent the ore to be comminuted. After the lumps of ore **50** to be comminuted come into contact through the feed opening **41** with the turning element **30**, the rotation of the turning element **30** causes the lumps of ore **50** to be accelerated radially outwards and in the rotational direction of the turning element **30**. For this purpose the two comminuting elements form a comminuting space, one or more accelerating elements being disposed on at least the turning element or the fixed element in order to bring about acceleration and corresponding comminution of the ore that has been fed in. By means of the rotation of the turning element **30** the ore to be comminuted is pulverised directly by the contact with the turning element **30** and also by the contact between lumps of ore which have already been partially comminuted and also by contact with the fixed element **40** in the comminuting space.

FIG. **8** shows the two comminuting elements of FIG. **7** in the opened up state together with ore **50** to be comminuted and pulverised ore **55** positioned by way of an example. The ore **50** to be comminuted is fed via the feed opening **41** through the fixed element **40** into the comminuting space between the two comminuting elements, as already described. Optionally, the turning element **30** has a ramp region **31** which has a rising incline from the start of the ramp **32** to the end of the ramp **33** and can be part of the comminuting space. By means of the rotation of the turning element **30** the ore **50** to be comminuted is already comminuted due to the rising ramp region **31**, as shown diagrammatically by the spherical particles of ore **51** and **52** which become smaller and smaller. The ramp region **31** co-operates here with an annular region **42** of the fixed element **40**. Next the ore is accelerated and pulverised by protrusions **35** which act as accelerating elements due to the rotation of the turning element **30** and which are arranged equal distances apart in the circumferential direction of the turning element **30** in FIG. **8**. The fixed element **40** can also have protrusions **45** which are arranged in the same way as the protrusions **35** of the turning element **30**. Corresponding recesses **36** are provided on the end face of the turning element **30** between the protrusions **35** of the turning element as part of the comminuting space. The protrusions **35** are in particular at a predetermined angle in the cross-over to the recesses **36** in order to accelerate the ore to be comminuted both in the radial direction according to the rotation and also in the axial direction of the axis of rotation of the turning element. In this way the ore to be comminuted is accelerated into the centre of the comminuting space and strikes against other accelerated ore elements here so that notional pulverisation is produced by the micro-impact.

Optionally, the fixed element **30** has corresponding recesses **46** between the protrusions **45** of the fixed element **30**. After the ore has been pulverised between the fixed element **40** and the turning element **30**, in particular by the acceleration by means of the protrusions **35**, the ramp region **31** and the protrusions **45** of the fixed element due to the rotation, the pulverised ore **45** passes into the intermediate space **60** between the two comminuting elements **30**, **40**.

As already described, the intermediate space **60** is formed by the variable distance between the two comminuting elements **30**, **40**, in addition to the variable distance star-shaped outlet notches **61** leading away from the axis of rotation of the turning element **30** also possibly being provided in the turning element **30**. Similarly, outlet notches **62** are provided equal distances apart in the fixed element **40**. As shown diagrammatically with regard to the turning element **30** in FIG. **8**, the pulverised ore **44** is discharged outwards through the outlet notches **61** and **62**. If the distance between the turning element **30** and the fixed element **40** is not provided, i.e. the two elements are substantially resting against one another, the pulverised ore **55** is substantially discharged outwards through the outlet notches **61** and **62**. The variable distance between the two comminuting elements can be adjusted in particular by a hydraulic unit, and preferably the fixed element **40** can be positioned variably in the axial direction in relation to the turning element **30** in order to be able to adjust the pulverisation as regards size and composition, in particular for a different ore.

According to a further embodiment the fixed element **30** or the turning element **40** or both comminuting elements can be separated from one another hydraulically in the axial direction for repair and fitting work. Alternatively, the comminuting elements can be moved apart from one another out of the operating position by means of a pivot movement of one of the two comminuting elements. In this way the accelerating elements **35**, for example, or other elements of the first comminuting means subjected to high mechanical stress can be worked on or replaced. Furthermore, this makes it possible for elements subjected to high mechanical stress within the first comminuting means or for example the accelerating elements of protrusions **35** to be able to be made of different materials and to be exchanged as required. In this way wearing parts within the comminuting space, such as for example the protrusions, can also be further adapted to different ores.

With regard to FIG. **6**, which shows a diagrammatically enlarged distance between the turning element **30** and the fixed element **40**, it is evident that with only a small distance the ore to be comminuted is thrown outwardly in the radial direction by the rotation and is contained by the housing **3** before the pulverised ore is discharged from the device **290** according to the invention via the outlet funnel **14**, for example by the force of gravity alone or additionally by a suction device or similar.

FIG. **9** shows a further embodiment of a fixed element **140** which has a feed opening **141** in the centre. The fixed element **140** is substantially identical to that of FIG. **8**, the fixed element **140** having outlet notches **162** set at an angle through which the pulverised ore is conveyed away to the outside.

In the form illustrated the fixed element **41** shown in FIG. **9** can also be used as a second turning element which can have a relative speed different to the turning element **30** illustrated in FIG. **8**.

The embodiment of a comminuting element shown in FIG. **9** has an angular region **144** which extends respectively to both sides from the accelerating element **143** to the recess **145**. However, these two angular regions **144** can also be provided on just one side of the accelerating element **143** depending on the rotational direction in order to accelerate the ore to be comminuted, depending on the direction of rotation of the comminuting element, both in the radial and in the axial direction in relation to the rotation of the comminuting element. In this way, together with the accelerating elements of the turning element **30** shown in FIG. **8**, particularly effective pulverisation can be produced, in particular when the accelerating elements of the turning element **30** also have an angular region which is congruent to the angular regions **144** of the comminuting element of FIG. **9** or are arranged substantially in a mirror image of one another.

FIG. **10** shows a cross-section of the fixed element **40** of FIG. **8**, the feed opening **41** having a funnel-shaped structure.

According to FIG. **11** a further embodiment of the comminuting elements according to the present invention is shown.

Alternatively to the comminuting elements according to FIGS. **7** to **10**, in FIGS. **11** to **13** further embodiments for co-operating comminuting elements are shown which can be arranged within the device according to the invention according to FIG. **6**.

In FIG. **11** a fixed element **240** and a rotating turning element **230** are shown, the ore **50** to be comminuted being fed via the feed opening **241** into the comminuting space between the fixed element **240** and the turning element **230**. As can be seen, furthermore, from FIG. **11**, the comminuting space between the fixed element **240** and the turning element **230** is formed such as to taper substantially conically outwards from the axis of rotation of the turning element **230**, by means of which on the one hand pulverisation of the ore is brought about. On the other hand it is evident from FIG. **12** that the turning element **230** has recesses **236** which are arranged equal distances apart around the axis of rotation of the turning element. By means of the cross-overs of the recess **236** arranged at an angle, these recesses **236** provide in particular acceleration and so pulverisation of the ore due to the rotation which provides a relative movement between the turning element **230** and the fixed element **240**.

FIG. **13** shows the fixed element **240** of FIG. **11** which co-operates with the turning element **230** of FIG. **12**. The fixed element **240** shows in the cross-section in FIG. **13** the feed opening **241**. Similarly to the turning element **230** the fixed element **240** has recesses **246** in the radial direction around the centre of the axis of rotation. In particular, the sloped regions of the recesses **236**, **246** of the turning element **230** and the fixed element **240** provide acceleration and comminution of the ore which is discharged outwards in pulverised form through the intermediate space **260** between the turning element **230** and the fixed element **240**.

According to the invention a method for comminuting ore and/or in particular slag is thus provided, the ore feed unit **1** being provided for feeding ore **50** to be comminuted to a first comminuting means. The first comminuting means is composed of at least two comminuting elements **30**, **40** that can be moved relative to each other, which elements form a comminuting space for the ore to be comminuted with each other such that by a relative moment in the form of a rotation of at least one of the two comminuting elements **30**, **40** the ore to be comminuted is pulverised in that one or more accelerating elements, in particular protrusions, are provided on at least one of the comminuting elements **30**, **40**, said accelerating elements being arranged in particular on the end face of one of the two comminuting elements **30**, **40**, and accelerating and comminuting the ore to be comminuted by the rotation of one of the two comminuting elements **30**, **40**. Between the two comminuting elements **30**, **40** and/or in at least one of the two comminuting elements an intermediate space **60** is provided through which during the rotation the pulverised ore is conveyed away outwards from the centre of the rotation or from the axis of rotation of the turning element and from the two comminuting elements **30**, **40**. The ore pulverised in this way between the two comminuting elements is discharged outwards through a outlet unit which is at least functionally connected to the intermediate space **60**.

Purely optionally, during the comminuting process water can also be fed into the comminuting chamber through a water inlet (not shown) or by feeding water through the ore feed unit. The water thus forms together with the ore during and after pulverisation a sludge-like compound, the water being conveyed away through the outlet unit together with the pulverised ore.

As already explained with regard to FIG. **8**, the ramp region **31** is particularly advantageous for the comminuting of slag because such a ramp region on the turning element provides pre-comminution of slag by means of the rotation of the turning element, protrusions and/or recesses being provided according to the invention in the comminuting elements after the ramp region in the direction of conveyance in order to pulverise the particularly brittle and hard slag.

For the person skilled in the art it is quite obvious that the number of protrusions on the two comminuting elements can respectively be equal, it also being possible, however, to provide a different number of accelerating elements on the two comminuting elements.

According to one embodiment (not shown), the two comminuting elements can rotate in opposite directions in order to increase the relative movement between the two comminuting elements. However, this leads to greater structural complexity, and is only to be implemented in special cases.

In particular, the shape of the comminuting chamber which is formed by the two comminuting elements can be of different designs, different types of accelerating element being able to be arranged in plate-shaped or wedge-shaped or some similar form by means of which the ore to be comminuted is accelerated and so pulverised between the two comminuting elements.

According to one embodiment (not shown), in addition to the comminuting between the two comminuting elements, a further comminuting chamber can also be provided which is provided independently of the two comminuting elements, but is however integrated into the device according to the invention.

A device according to the invention and a method according to the invention for comminuting ore and/or in particular slag are thus described which comprise an ore feed unit for feeding ore to be comminuted to a first comminuting means, the first comminuting means being composed of at least two comminuting elements that can be moved relative to each other, which elements form at least one comminuting space for the ore to be comminuted with each other such that, by a relative movement in the form of a rotation of at least one of the two comminuting elements the ore to be comminuted is pulverised in that one or more accelerating elements, in particular protrusions, are provided on at least one of the comminuting elements, said accelerating elements being arranged in particular on the end face of at least one of the two comminuting elements and accelerating and comminuting the ore to be comminuted by the rotation of one of the two comminuting elements, and there being provided between the two comminuting elements and/or in at least one of the two comminuting elements an intermediate space through which during the rotation the pulverised ore can be conveyed away outwards from the centre of the rotation and from the two comminuting elements, and an outlet unit, in particular an outlet unit, being provided which is connected to the housing of the device through which the pulverised ore is discharged.

An exploded view of the device **290** according to the invention is depicted in FIG. **14**. This illustration shows, that the device **290** comprises in the region of a first comminuting means **300** a feeding means **1**, in particular a feeding funnel **1**, by means of which ore to be processed is conductable into housing **3** to the first comminuting means **300**. The housing **3** is preferably by means of two plate-like formed feets **2**, **6** with respect to the underground positioned respectively coupled with a preferably on the underside of the housing **3** arranged frame element **305**. Housing **3** of the first comminuting means **300** preferably has an opening **4**, in particular a suction opening **4** for sucking off of already comminuted ore. Further, underneath housing **3** respectively in the lower region of housing **3**, that means preferably in the region underneath the first comminuting means **300** and/or underneath the second comminuting means **301**, an outlet unit **14** (cf. FIG. **17**) is formed.

Reference number **340** preferably characterizes a hydraulic means (cf. FIG. **20** a/b).

The second comminuting means **301** is preferably formed laterally beside the first comminuting means **300**. The first comminuting means **300** and the second comminuting means **301** are arranged on the same frame element **305**. A wall of housing **306** of housing **3** is preferably on a first site coupled with the first comminuting means **300** and on another side with the second comminuting means **301**. The wall of the housing **306** preferably comprises multiple fixing locations **354**, **381** for arranging, receiving and/or fixing of a first means **302** for fixing and/or mounting of a preferably as mill ring **344** formed rotation body, a second means **303** for fixing and/or mounting of the mill ring **344** and a third means **304** for fixing and/or mounting of the mill ring **344**. Mill ring **344** is due to movement means **302**, **303** and **304** preferably movable mounted and actuatable. Further, mill ring **344** surrounds in radial direction preferably at least one further rotation body **345** and particular preferably at least or exactly two rotation bodies **345**, **380**, which are particular preferably formed as drum-like bodies. Further, in the wall of the housing **306** preferably an opening **382** is formed. The first opening **382** particular preferably serves for putting through the shaft, which is provided for actuating comminuting element **30**.

The first means **302** and the second means **303** are preferably formed identical and in vertical direction preferably arranged underneath a centre of the mill ring **344**. Means **302**, **303** can also be considered as axes or movable shafts **371**,**313**. Each one of the first means **302** and the second means **303** preferably comprises an element for the application of force, in particular a drive wheel **367**. The actuating elements **367** are preferably mechanically coupled with each other and therefore at the same time respectively synchronous movable respectively actuatable. In axial direction are preferably joined to the drive wheel **367** a disc element **364**, a fixing body **366**, a fence element **36**, bearings and/or one or multiple receiving bushs, by means of which the axes respectively shafts **371**, **313** are preferably directable into a functional connection.

A drive wheel **367** of a means **302**, **303** is preferably immediately or mediatly connected with a further actuating element **368**, in particular a gear for transferring actuation forces. Gear **368** is preferably connected via an endless element **369**, in particular a chain or a belt, with a further actuating element, in particular a further gear **368**, which is preferably directly arranged at an actuating means, in particular a motor **370**. However, it is also conceivable, that motor **370** directly interacts with one of the drive wheels **367** respectively is arranged thereon.

The third means for fixing and/or transmission of force **304**, which is preferably considerable as upper axis respectively shaft **357**, is preferably arranged above the centre of mill ring **233** and particular preferable arranged in vertical direction exactly above the centre of mill ring **344**. The third means **304** preferably has a disc element **365**, a fixing body **363**, an inner cover element **362**, a bolt nut **360**, a washer **359**, bearings **358** and/or one or more receiving bushs **355** by means of which the axis respectively shaft **367** is preferably directable into a functional connection with mill ring **344**.

The first means **302**, the second means **303** and/or the third means **304** are preferably essentially or exactly aligned in parallel with respect to each other, wherein preferably at least one of those means **302**, **303**, **304** is also essentially or exactly aligned in parallel to the rotation axis of a comminuting element.

Further due to reference number **307** a forth means for fixing and/or transmitting of forces is characterized. The forth means **307** preferably serves for alignment respectively holding of the rotation body **345**, **380** with respect to mill ring **344**. However, it is also conceivable that the forth means **307** comprises an actuation means for active actuation of one respectively the rotation bodies **345**, **380** receptively is coupled with such an actuating means. The forth means **307** preferably can be considered as axis or shaft **351** and preferably comprises an outer cover element **354**, a fixing means **366**, an inner cover element **352**, a spacing element **348** for receiving and/or spacing the axes **347**, bearing cover elements **348**, axes **347** and/or roller bearings **346**. The rotation bodies **345**, **380** are therefore rotatable mounted by bearings **346**.

FIG. **15** shows a perspective illustration of parts of the second comminuting means **301**. According to this illustration the second comminuting means **301** has a rotation body formed as mill ring **34**, which at least sectionally and preferably completely surrounds radially two further rotation bodies **345**, **380**, which are formed as drum-like mill elements respectively mill-drums. Mill ring **344** and mill drums **345**, **380** have axially preferably essentially the same length, wherein it is also conceivable, that mill drums **345**, **380** implemented axially longer as mill ring **344** respectively vice versa. Mill drums **345**, **380** preferably comprise an outer surface **383**, which is preferably formed spherically, in particular starting from its essentially axial center to its axial ends conical tapered. The inner surface **383** of mill ring **344** is preferably formed cylindrical, wherein it is also conceivable that it is formed negative or essentially negative with respect to the outer surface **383** of mill drums **345**, **380** The outer surface **384** of mill ring **344** is preferably formed cylindrical. The outer surface **384** of mill ring **344** are contacting preferably exactly three means **302**, **303**, **304** for fixing and/or force transmission, in particular respectively by means of element **55** for guiding mill ring **344**, preferably in line contact and particular preferably in areal contact.

Reference number **348** preferably characterizes a bearing cover, which preferably covers at least sectionally radially the drum body of mill drum **380** and the bearing, which preferably consists of preferably at least or exactly two roller bearing **346** (cf. FIG. **14**), in particular covers in such a manner, that the bearing is protected against the entering of ore powder.

The rotation axes of both mill drums **344**, **380** are preferably arranged spaced apart by means of a spacing

element **349**. The spacing element **349** is preferably formed as strut shaped, in particular plate shaped, receiving element, in particular out of metal. Beside the mill drums **345**, **380** a fixing body **366** is preferably also arranged at the spacing element **349** respectively coupled with the spacing element **349**. Hereby the fixing body **366** can be provided for one-sided attachment of mill drum units **345**, **380**, **348**, **349** at a housing part (not shown), in particular a further wall of the housing. However, it is also conceivable that fixing body **366** is formed as actuating unit **366** and serves for active actuating of mill drums **344**, **380**.

The first means for fixing and force transmission **302** and the second means for fixing and force transmission **303** have gears **367**, which are connected with each other by means of a chain **360**. It is further obvious, that the second means for fixing and force transmission **303** is also equipped with a round disc-like force transmission plate **368**, which is radial formed for receiving a belt **372**, by means of which the second means for fixing and force transmission **302** is coupled with a further round force transmission plate **368**, which again is connected with an actuating means **370**, in particular a motor for operating the second comminuting means **301**.

A cross-sectional view through the ore comminuting device **290** according to the invention is shown in FIG. **16***b*. The device housing **3** is gatherable from this illustration, which is held by means of feats **6** with respect to the underground respectively a receiving rack (cf. FIG. **19** or FIG. **20***a/b*). Housing **3** preferably surrounds the second comminuting means **301** in circumferential direction completely. On the inner surface of housing **3** respectively on the surface side facing the second comminuting means of the housing preferably multiple holding means are arranged, in particular exactly three holding means namely a first holding means **402**, a second holding means **403** and a third holding means **404**. The holding means **402**, **403**, **404** preferably serve for positioning respectively holding of actuating and/or guiding elements **355**. The actuating and/or guiding elements **355** are preferably rollers, which are arranged rotatable at the holding means **402**, **403**, **404**. At least one of the actuating and/or guiding elements **355** is preferably actuated by means of a motor. Particular preferably two or all actuating and/or guiding elements **355** are actuated, in particular by means of a motor or by means of a respective motor. The actuating and/or guiding elements **355** serve for actuating and guiding of mill ring **344**. Mill ring **344** is preferably adjacent to the wall of housing **406**. The wall of housing **406** preferably comprises a central opening **382**, which is provided for through putting of an actuating means, in particular a shaft, for actuating the first comminuting means **300**, in particular of comminuting element **30** (cf. FIG. **6** and FIG. **17**). Further, a feeding means **408** is formed within the wall of housing **406** respectively feeding means **408** is preferably designed tubular and extends through wall **406**. The feeding means **408** preferably serves for feeding of material already comminuted by the first comminuting means **300**. The feeding means **408** preferably extends in such a manner inside housing **3** respectively into a region surrounded by mill ring **344**, that the material fed by means of the feeding means **408** is inserted before the first mill drum **345**. Mill ring **344** preferably rotates in the direction characterized with reference sign R, whereby the material introduced before the first mill drum **345** is fed between mill ring **344** and mill drum **345**. The material is further comminuted respectively pulverized due to the interaction of mill ring **344** and mill drum **345**. Further, a second mill drum **380** is shown, it is therefore conceivable that multiple mill drums **345**, **380** are installed. It is preferably conceivable that any number of mill drums **345**, **380**, in particular exactly, more or less than one, two, three, four or five mill drums, are installed. The individual mill drums **345**, **380** are preferably rotatable and particular preferably active actuated by means of an actuating means. Further it is conceivable that mill drums **345**, **380** are rotated respectively actuated only passive, that means as a result of a rotation of mill ring **344**. The mill drums **345**, **380** are preferably arranged at the wall of housing **406** by means of spacing elements **349** for receiving the mill drums **345**, **380** via coupling locations **412**. It is hereby conceivable that the positions of mill drums **345**, **380** are adjustable respectively modifiable by means of spacing elements **349**. A distance, in particular a maximum distance, of the outer mill drum surface to the inner mill ring surface is preferably adjustable.

It is further conceivable, that mill drums **345**, **380** or one of those mill drums **345**, **380** is spring loaded respectively is pressed againsted the mill ring respectively is pretensioned.

A ore comminuting device **290** according the invention is shown in FIG. **17** broadend with respect to FIG. **6***a* by the second comminuting means **301**. The ore comminuting device **290** comprises a feeding funnel **1** via which coarse material to be comminuted is inserted into the device. The material is comminuted by means of the first comminuting means **300**, in particular by means of interacting elements **30**, **40**, that means comminuting element **30** and fixing element **40**. The comminuted material parts are moved outwardly from the region between the elements **30**, **40**, in particular by means of gravitation, and get to a funnel **14**. The elements **30**, **40** are preferably arranged with respect to each other in a distance of essentially, exactly or at most 7 cm and further preferred in a distance of essentially, exactly or at most 5 cm and particular preferred in a distance of essentially, exactly or at most 3.5 cm. Hereby it is conceivable that the distance between the elements **30**, **40** is adjustable, in particular variable. The distance between elements **30**, **40** can be particular preferably stepless or in predefined steps adjusted. Funnel **14** conducts the comminuted material according to arrow T**1** via a pumping means **410** in a separator respectively in a separating unit **413**. Separator **413** divides, in particular cyclone-like, sufficiently crushed material parts from not sufficiently crushed material parts. Not sufficiently crushed material parts, which are separated from the sufficiently crushed material parts by separator **413**, are outputted from the separator **413** via a first outlet opening **414** or a junction and according to the feeding line characterized by reference sign T**2** fed to an inserting means **408** (cf. FIG. **16**). Inserting means **408** is preferably arranged in the region of wall **406** and serves for inserting of material parts to be further comminuted into the second comminuting means **301**. Additionally or alternatively it is also conceivable that the material parts to be further comminuted are again fed to the first comminuting means **300**. Reference number **416** characterizes a second outlet opening respectively a further junction. By means of the second outlet opening **416** respectively by means of the further junction sufficiently comminuted ore is channeled of respectively conveyed according to feeding line T**3** out of the region of device **290**, wherein the ore is preferably immediately fed respectively conducted to a floating means. It is further conceivable that separator **413** comprises three outlet unit and assignes the comminuted material to three ranges of material size, wherein the already sufficiently comminuted material is fed according to T**3** and the not sufficiently comminuted material is separated into a coarse portion and a fine portion. Then, the coarse portion is again feedable to the first comminuting means **300** and the fine portion is feedable to the second comminuting means **301**, in particular according to T**2**.

The sufficiently comminuted, in particular pulverized, material parts are discharged from the ore comminuting device according to the arrow characterized by reference sign T**3** and particular preferable immediately fed to a floating means.

It is gatherable from this illustration that at least two shafts **357**, **371** are provided. Shafts **357**, **371** serve for actuation of the elements for guiding and/or actuating **355**. The individual shafts **357**, **371** are preferably connected with actuating means **304**. Further a third shaft (cf. FIG. **14**) for actuating a third element for guiding and/or actuating **355** (cf. FIG. **15**) is particular preferably provided.

Further, mill drums **345**, **380** are illustrated, which are surrounded in circumferential direction by the mill ring.

Further, reference number **504** characterizes a spring means, which can be e.g. formed as mechanical pressure spring respectively coil spring, gas spring or as hydraulic spring. The spring means **504** causes that a force of several tons is axially applied to shaft **21** and therewith the comminuting element **30**. This means that an axial sliding of shaft **21** in X-direction happens only then, if e.g. as a result of a material jam forces are generated between comminuting elements **30**, **40**, which are directed into X-direction and exceed the spring force. The spring means **504** therefore causes in beneficial manner, that shaft **21** and comminuting elements **30**, **40** are in X-direction only subjected to a predefined respectively adjusted maximum force, whereby those elements are protected against damage. The sliding path S1 of shaft **21** due to a displacement of spring means **504** preferably is in the range of a few respectively several millimeters up to a few respectively several centimeters.

Further is conceivable that the spring force is adjustable respectively predefinable in such a manner, that defined ore particle sizes are generatable. The smaller the spring force, the larger are the resulting sizes of the ore particles.

The spring force is preferably stepless respectively continuously or in steps adjustable.

Reference numbers **506** and **508** characterize roller bearings, by means of which shaft **21** is preferably mounted. Roller bearings **506** are preferably formed as ball bearings and roller bearings **508** are preferably formed as cone bearings or needle bearings.

FIG. **18** shows the embodiment of FIG. **17** in an open configuration. In this configuration preferably at least the comminuting element **30** and preferably the complete internal space of device **290** is accessible to a human for maintenance work. The housing cover **420** is thereby moved by means of an actuator **434** respectively by means of multiple actuators, in particular exactly two actuators **434**, of a hydraulic means (cf. FIG. **21***a/b*) into the opened position.

A transportation means **386** is shown in FIG. **19***a* in a top view, on which a comminuting device **290** according to the invention is arranged. Transportation means **386** is preferably formed as trailer, which can be pulled by a motor driven vehicle. Transportation means **386** therefore comprises a frame **388** on which the comminuting means **290** is preferably permanently arranged. However it is also conceivable that comminuting means **290** is detachable coupled with transportation means **386**. On frame **388** are preferably at least or exactly two wheels arranged for each axis. In the illustrated embodiment transportation means **386** comprises exactly one axis, wherein it is conceivable that it comprises multiple, in particular two or three axes. Transportation means **386** is coupleable via coupling location **392** with a vehicle or a further trailor.

In FIG. **19***b* a sideview of the illustration shown in FIG. **19***a* is depicted.

In FIG. **20** a comminuting device **290** according to the invention is arranged on a pedestal **393**. However, in place of pedestal **393** comminuting device **290** can be arranged alternatively on a scaffold or a platform. The arrangement shown in FIG. **20** is beneficial since the outputting region **394** from which the comminuted material is outputted is easily accessible because of the distance between comminuting means **290** and the underground.

Further, the actuating means respectively the motors are characterized by reference numbers **450**, **452**, by means of which rotation ring body **344** (cf. FIG. **15**) is actuatable.

FIG. **21***a* shows the device **290** according to the invention in a closed configuration. In this closed configuration the housing cover **420**, which is in contact with the feeding funnel **1**, lies, in particular sealingly, on the housing **3**. The housing cover **420** is preferably holded by means of a closing means **430**, which is particular preferably formed as hydraulic means, and preferably pressed against housing **3**. The hydraulic means **430** preferably comprises a stator **432**, which is particular preferably arranged in the region of housing **3** or on housing **3**. Stator **430** is preferably coupled with the actuator **434** in such a manner, that it is slideable in the direction of extention of the rotation axis of comminuting element **30**. On both sides of housing **3** a hydraulic means **430** is preferably arranged. Further, it is conceivable that the mentioned hydraulic means are also arranged in the region of the upper and lower wall regions of housing **3**. It is also conceivable that more than two, in particular three or four, hydraulic means **430** are provided, in particular in the upper and lower housing region and in the lateral housing regions. In case of multiple hydraulic means **430** these are preferably simultaneously, in particular via a control means, selectable. Actuator **434** is preferably coupled respectively connected with housing cover **420** by means of an actuator-housing-cover-coupling-location **436**.

Device **290** is illustrated in FIG. **21***b* in an open respectively opened configuration. The open respectively opened configuration is thereby characterized that housing cover **420** is at least sectionally removed respectively spaced apart from housing **3**. Such spacing apart can take place as shown, that means housing cover **420** can in total be spaced apart from housing **3** about a preferably defined path. Spacing apart preferably takes place by means of one or multiple hydraulic means **432**. However it is also conceivable that housing cover **420** lies on the one hand side on the housing **3** and is pivoted by means of the locking means respectively holding means **430** around a contact point.

The feeding funnel **1** and the comminuting element **40** are preferably arranged at housing cover **420**. By means of feeding funnel **1** the ore to be feeded is preferably funnelable through housing cover **420** and through comminuting element **40** into the closed housing **3** (cf. FIG. **21***a*).

Further the illustration of FIG. **21***b* show as a human characterized by reference number **500**. It can be further gathered from this illustration, that by means of hydraulic means **432** the housing cover **420** with the thereon arranged means, in particular the comminuting element **40**, is particular preferably movable that far, that a human **500** can access the device through opening **502** resulting from the movement of the housing cover respectively can maintain single or all components therein. As maintenance work wear elements, like e.g. the ramp region **31**, the protrusions **35**, the protrusions **45** of both comminuting elements **30**, **40**, can be exchanged.

Hydraulic means **432** can serve additionally or alternatively as spring means for variable mounting of comminuting element **40**.

The device according to the invention has procedural benefits in dry and/or wet processing. In this context a process independence from water is important. The device according to the invention works dry as well as wett—a benefit, which the process chain of crushers and mills has to differentiate according to the function. Further crushes the Micro Impact Mill also slag or a mixture of slag and ore material, which overcharges the crushing technique of classic facilities due to the hardness of the material.

It is further beneficial, that this device can process rocks and/or slag. Even bricks of furnaces do not affect it. In view of the scope of performance the device according to the invention can even replace the overall process chain consisting of crushers and ball mills. Rocks preferably with up to 80 cm, further preferably with up to 50 cm and particular preferably with up to 40 cm are directly processed suitable for flotation in one process step. This is faced with multiple crushing stages with crushers until the ball mills are in charge.

Due to the micro impact in particular only small wear takes place in the device according to the invention, that means due to the repetitive collision of ore differently accelerated, whereby the mechanical elements are only subjected to small load, wherein also no further loose milling elements or iron balls have to be used.

Furthermore, the device according to the invention and the method according to the invention enables that slag itself or together with ore material can be comminuted and pulverized, since due to the small dimensions of the comminuting space as well as the relative small dimensioned comminuting elements with a respective rotation high forces are applied on the ore material to be comminuted respectively the slag to be comminuted and thus an effective comminuting takes place. Due to the rotation, which comprises because of the dimensions 100 up to more or less 2000 revolutions per minute of a comminuting element, also slag can be pulverized in an effective manner, which is very brittle and comprises a hard structure.

With the device according to the invention the productivity of resources as well as the conserving of resources can be enhanced. With this innovation there is no need for pre-crushing with crushers and mills—in a very energy efficiency and ecological manner. This innovative device is further beneficial, because it connects energy and resource efficiency and simultaneously provides a totally new human-machine-cooperation completely without silicosis and noise-induced deafness.

LIST OF REFERENCE NUMBERS

1 Feeding funnel
2 Foot
3 Housing
4 Suction opening
6 Foot
8 Motor
9 Belt pulley
10 Belt
11 Drive roller
14 Outlet funnel
15 Control flap

21 Shaft
30 Comminuting element
31 Ramp region
33 Ramp end
35 Protrusion
36 Recess
40 Fix element
41 Feeding opening
42 Reing region
45 Protrusion
46 Recess
50 Ore clump
51 Ore particle
52 Ore particle
55 Comminuted ore
60 Intermediate space
61 Outlet notches
62 Outlet notches
140 Fix element
141 Fix element
143 Acceleration element
144 Angular region
145 Recess
162 Outlet notches
230 Rotation element
236 Recess
240 Fix element
241 Feeding opening
260 Intermediate space
290 Comminuting device
300 First comminuting means
301 Second comminuting means
302 First means for fixing and force transmission
303 Second means for fixing and force transmission
304 Third means for fixing and force transmission
305 Frame element
306 Wall of the housing
307 Forth means for fixing and/or force transmission
313 First lower shaft for fixing and/or actuating of the mill ring
344 Mill ring
345 First Mill drum
346 Roller bearing
347 Shaft
348 roller bearing covering element
349 Spacing element for receiving and spacing apart of shaft **347**
350 Fixing of the element for spacing apart
351 Shaft
352 Inner roller bearing covering element
354 Fixing position
355 Element for guiding and/or actuating of the mill ring
356 Means for securing a shaft
357 Upper shaft for fixing and/or actuating the mill ring (respectively the axis)
358 Roller bearing for mounting the mill drum
359 Washer
360 bolt nut
361 Stop collar for fixing the mill ring
362 Inner cover element
363 Upper fixing body for fixing the mill ring
364 Disc element for fixing of a lower axis supporting the mill ring
365 Disc element for fixing an upper shaft supporting the mill ring
366 Lower fixing body for fixing the mill ring
367 Drive wheel
368 Round disc-like force transmission disc
369 Drive chain
370 Motor
371 Second lower shaft for fixing and/or actuating the mill ring
372 Belt
380 Second mill drum
381 Fixing position
382 Opening
383 Outer surface of the mill drum
384 Outer surface of the mill ring
385 Inner surface of the mill ring
386 Transportation means
388 Frame
390 Wheels
392 Coupling location
393 Rack
394 Outputting region
402 First holding means
403 Second holding means
404 Third holding means
406 Wall
408 Feeding means
410 Pumping means
412 Coupling location at the wall
413 Separating means
414 First outlet opening in the separator
416 Second outlet opening in the separator
419 Conduit section
420 Housing cover
430 Hydraulic means
432 Stator
434 Actuator
436 Actuator-Housing-Cover-Coupling
450 First additional actuator
452 Second additional actuator
500 Human
502 Opening
504 Spring means
506 Roller bearing
508 Roller bearing
520 Feeding connection
521 Axial end of the shaft
R Direction of rotation of mill ring
S1 Sliding path
T1 First transportation direction
T2 Second transportation direction
T3 Third transportation direction
X Direction

The invention claimed is:

1. A device for comminuting ore and/or slag comprising:

an ore feed unit for feeding ore to a first comminuting means;

wherein the first comminuting means includes at least two comminuting elements that are moveable relative to each other and form at least one comminuting space for the ore to be comminuted such that, by a relative movement in the form of a rotation around a rotational axis of at least one of the comminuting elements, the ore is pulverised using one or more accelerating elements having protrusions on at least one of the comminuting elements, the accelerating elements being arranged on an end face of one of the comminuting elements and accelerating and comminuting the ore to by the rotation of one of the comminuting elements;

an intermediate space being provided between the comminuting elements and/or in at least one of the comminuting elements through which the pulverised ore is conveyed during the rotation outwards from a center of the rotation and from the comminuting elements;

wherein at least one of the comminuting elements includes a functional connection with a spring means, the spring means being formed to mount at least one of the comminuting elements in functional connection so as to be variable in a direction of the other comminuting element, wherein at least one of the comminuting elements is arranged on a shaft for actuating at least one of the comminuting elements;

wherein the spring means is pretensioned through direct coupling with the shaft or at least one of the comminuting elements;

wherein the shaft and at least one of the comminuting elements are arranged to be slideable opposite to the spring force of the spring means;

a spiral shaped and ramped region is configured relative to at least one of the comminuting elements to provide a pressure application on the ore to be comminuted; and

wherein at least one of the comminuting elements is arranged in a direction of extension of the rotational axis at a housing of the device to open and close a housing cover, wherein the housing cover is moveable with respect to the device and wherein at least one of the comminuting elements is pressed against the other comminuting element with the spring means when the spring means connects the housing cover with the device.

2. The device according to claim **1** wherein a sliding of the shaft and at least one of the comminuting elements takes place in dependency of the pretension of the spring means such that a deflection of the spring means results during operation of the first comminuting means due to a deflection force generated between the comminuting elements and directed opposite to a contact pressing force resulting from the spring force in the event the deflection force exceeds the contact pressing force.

3. The device according to claim **1** wherein the spring means comprises a mechanical spring means including a spiral spring, a pneumatic spring and/or a hydraulic spring.

4. The device according to claim **3** wherein the spring means has multiple suspension means with at least one suspension means being arranged in such a manner to push at least one of the comminuting elements coupled with the shaft into the direction of the other comminuting element.

5. The device according to claim **1** wherein the shaft is mounted in the housing of the device using roller bearings and is coupled with a actuating means for rotating the shaft and at least one of the comminuting elements.

6. The device according to claim **1** wherein the spring means is arranged in an end region of the shaft, the end region being spaced apart from a second end region of the shaft having one of the comminuting elements.

7. The device according to claim **1** wherein a spring constant of the spring means, a sliding path of the comminuting element and/or a deflection path of the spring means are adjustable or exchangeable.

8. A method for comminuting ore and/or slag comprising:

providing ore to an ore feed unit for feeding to a first comminuting means, wherein the first comminuting means includes at least comminuting elements that are moveable relative to each other and form at least one comminuting space for the ore to be comminuted;

providing a relative movement in the form of a rotation around a rotational axis of at least one of the comminuting elements to pulverise the ore such that one or more accelerating elements, including protrusions, are used on at least one of the comminuting elements, the accelerating elements being arranged in particular on an end face of one of the comminuting elements and accelerating and comminuting the ore to by the rotation of one of the comminuting elements;

conveying during the rotation the pulverised ore outwards from a center of the rotation and from the comminuting elements through an intermediate space between the comminuting elements and/or in at least one of the comminuting elements;

mounting a spring means in a functional connection with at least one of the comminuting elements with variability in a direction of the other comminuting element;

arranging at least one of the comminuting elements on a shaft used for actuating at least one of the comminuting element;

coupling with pretension the spring means directly with the shaft or at least one of the comminuting elements;

arranging the shaft and at least one of the comminuting elements to be slideable opposite to a spring force of the spring means;

arranging a spiral shaped and ramped region relative to at least one of the comminuting elements to provide a pressure application on the ore to be comminuted; and

arranging at least one of the comminuting elements in a direction of extension of the rotational axis of a housing of the device to open and close a housing cover, the housing cover being moveable with respect to the device such that at least one of the comminuting elements is pressed against the other comminuting element by means of the spring means when the spring means connects the housing cover with the device.

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