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Leone

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(54) **SYSTEMS AND METHODS FOR IMPROVING TORQUE RESPONSE OF AN ENGINE**

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(58) **Field of Classification Search**

None

See application file for complete search history.

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This patent is subject to a terminal disclaimer.

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B60W 10/06 (2006.01)

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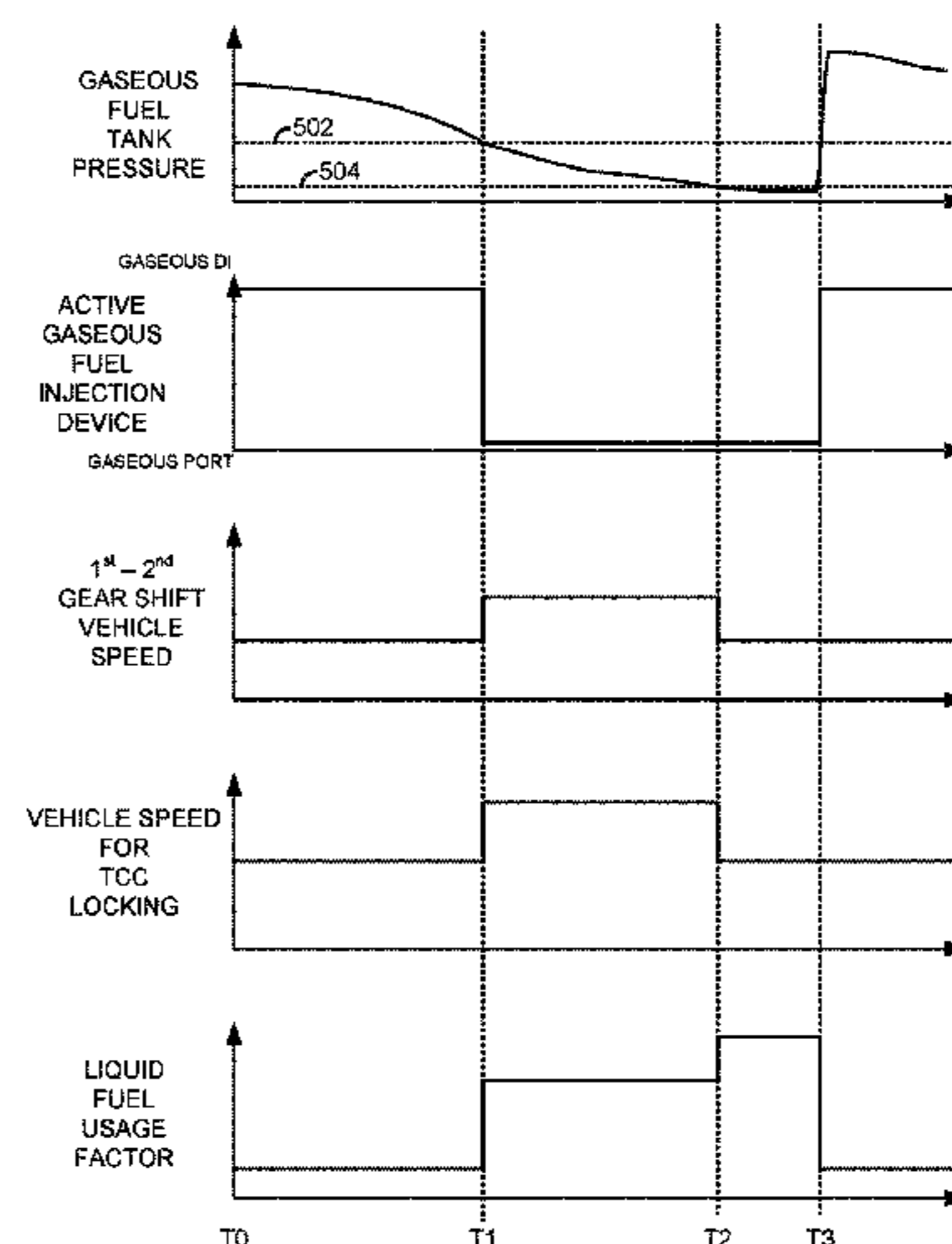
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(57) **ABSTRACT**

Systems and methods for improving engine torque response are presented. In one example, engine idle speed is increased to shorten engine torque response based on engine operating conditions. The methods and systems may be useful for operating an engine that is supplied a gaseous fuel.

20 Claims, 8 Drawing Sheets



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 (2013.01); *F02D 2200/0602* (2013.01); *Y02T*
10/32 (2013.01)

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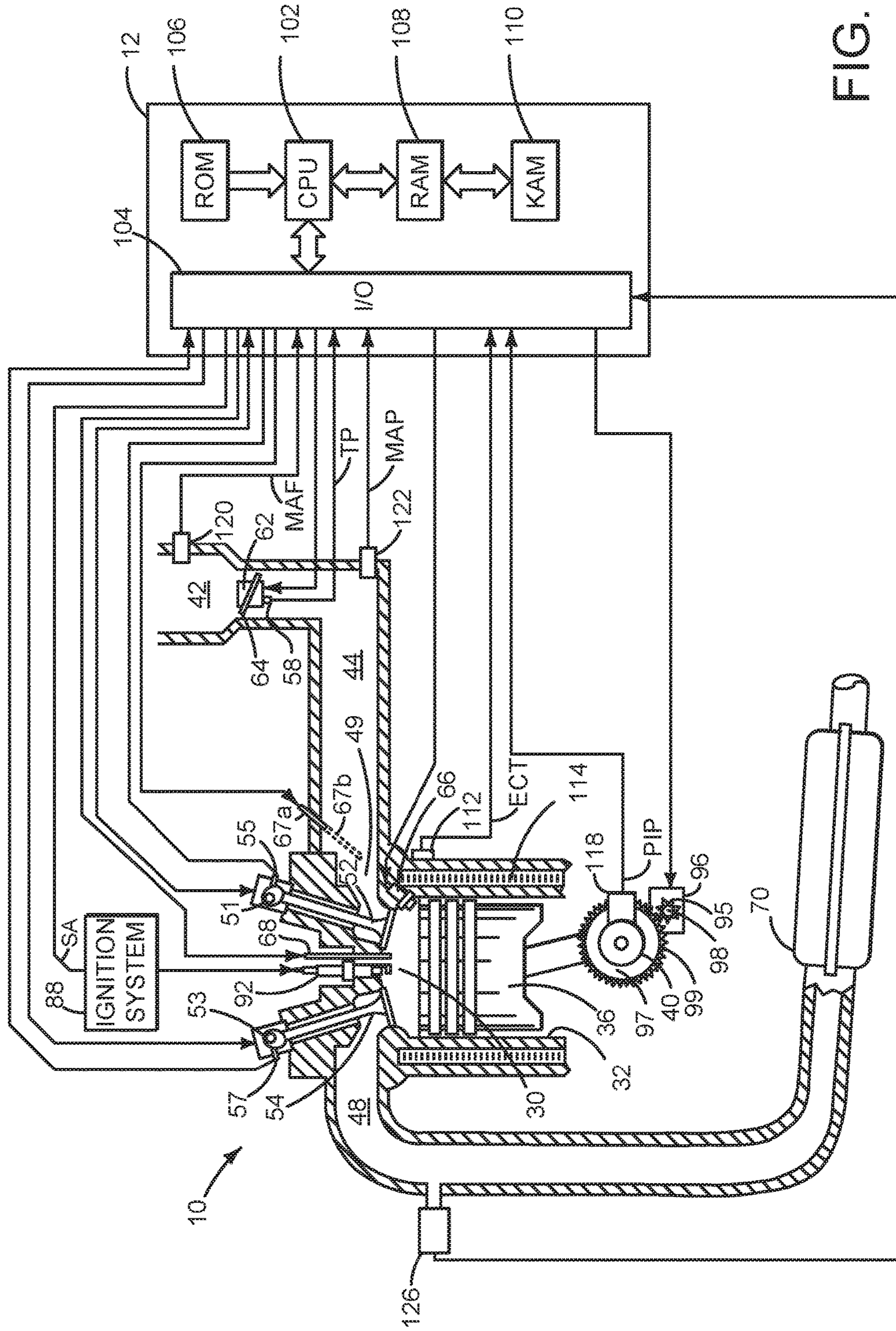


FIG. 1

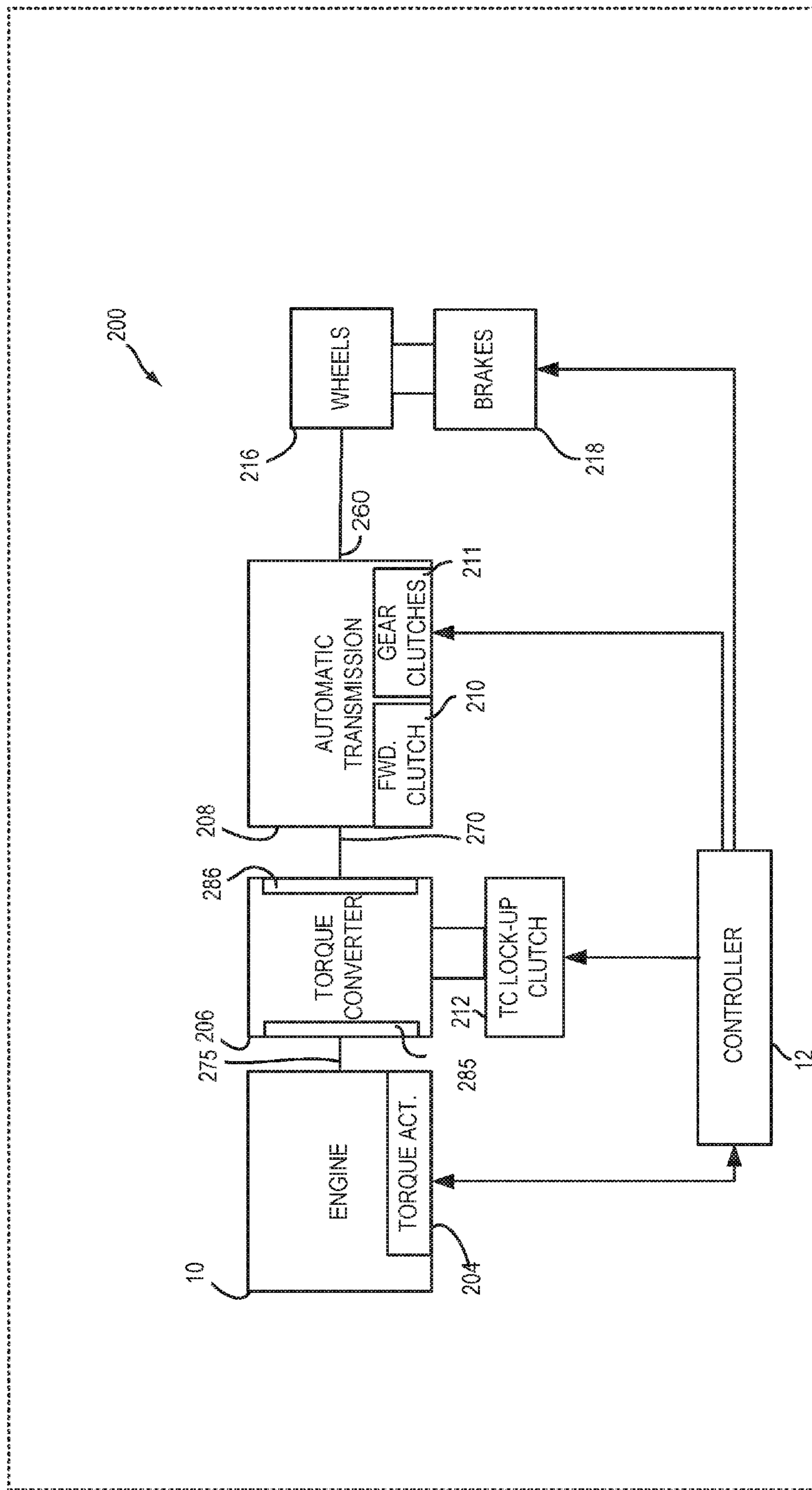


FIG. 2 290

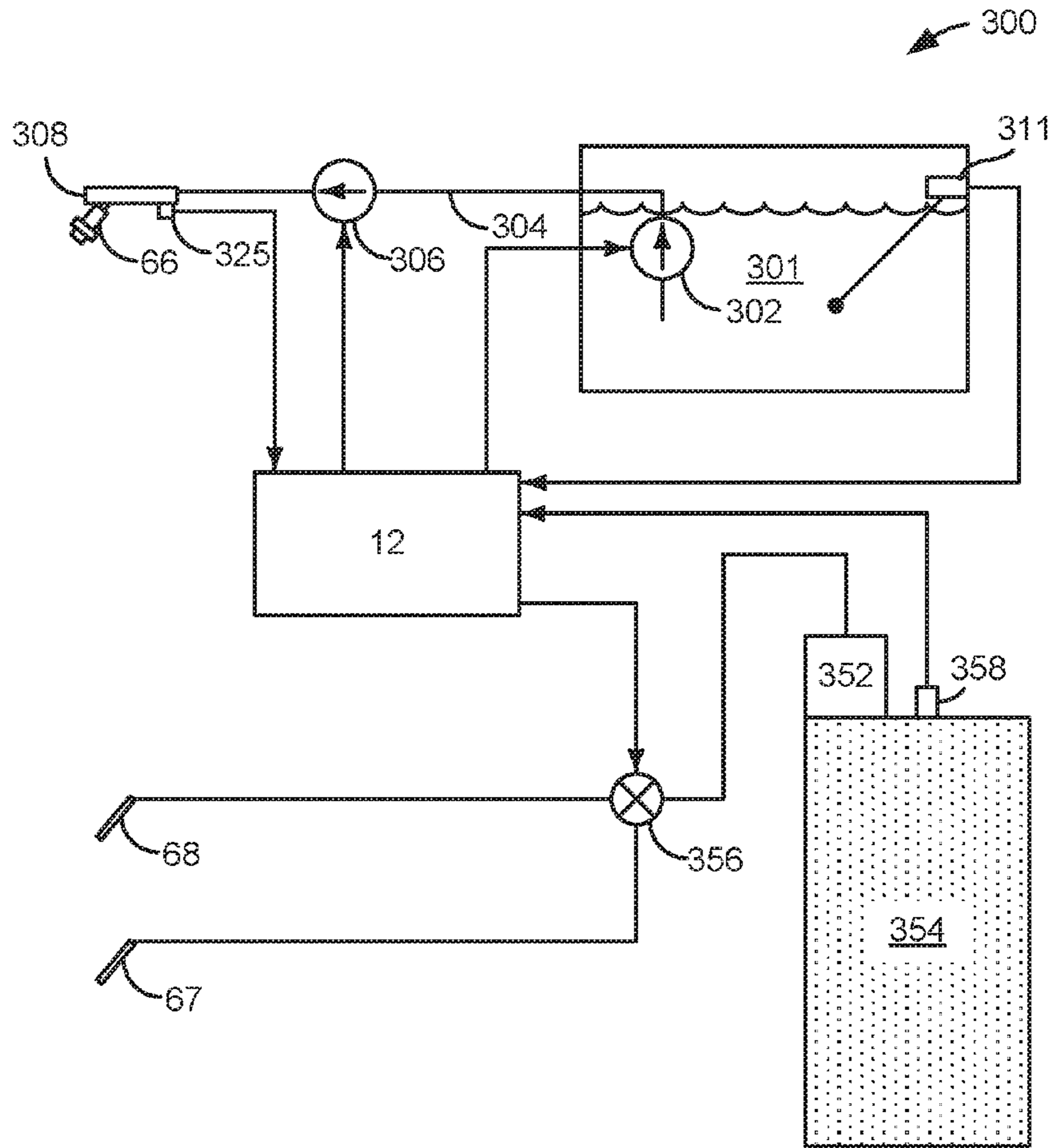


FIG. 3

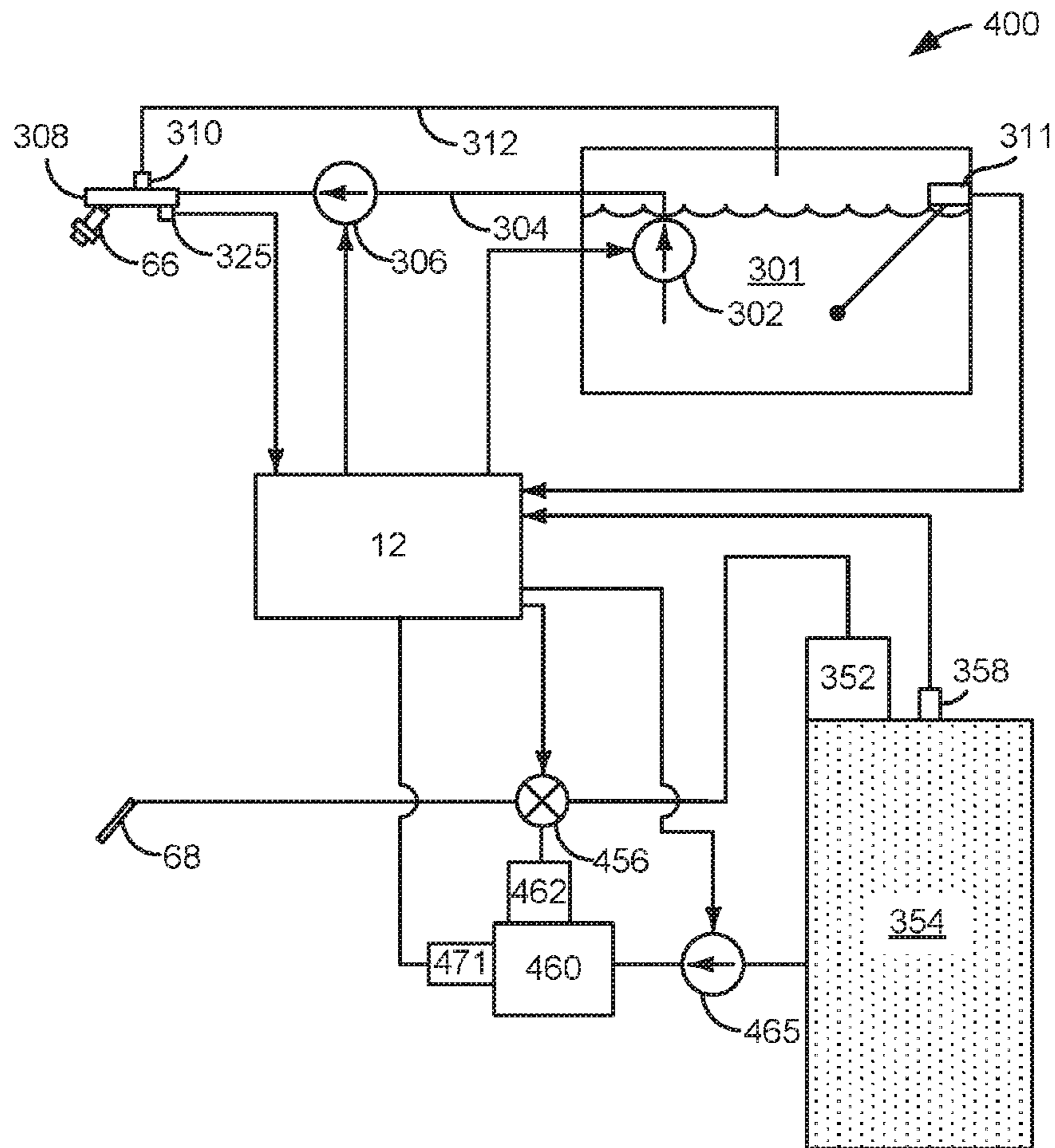


FIG. 4

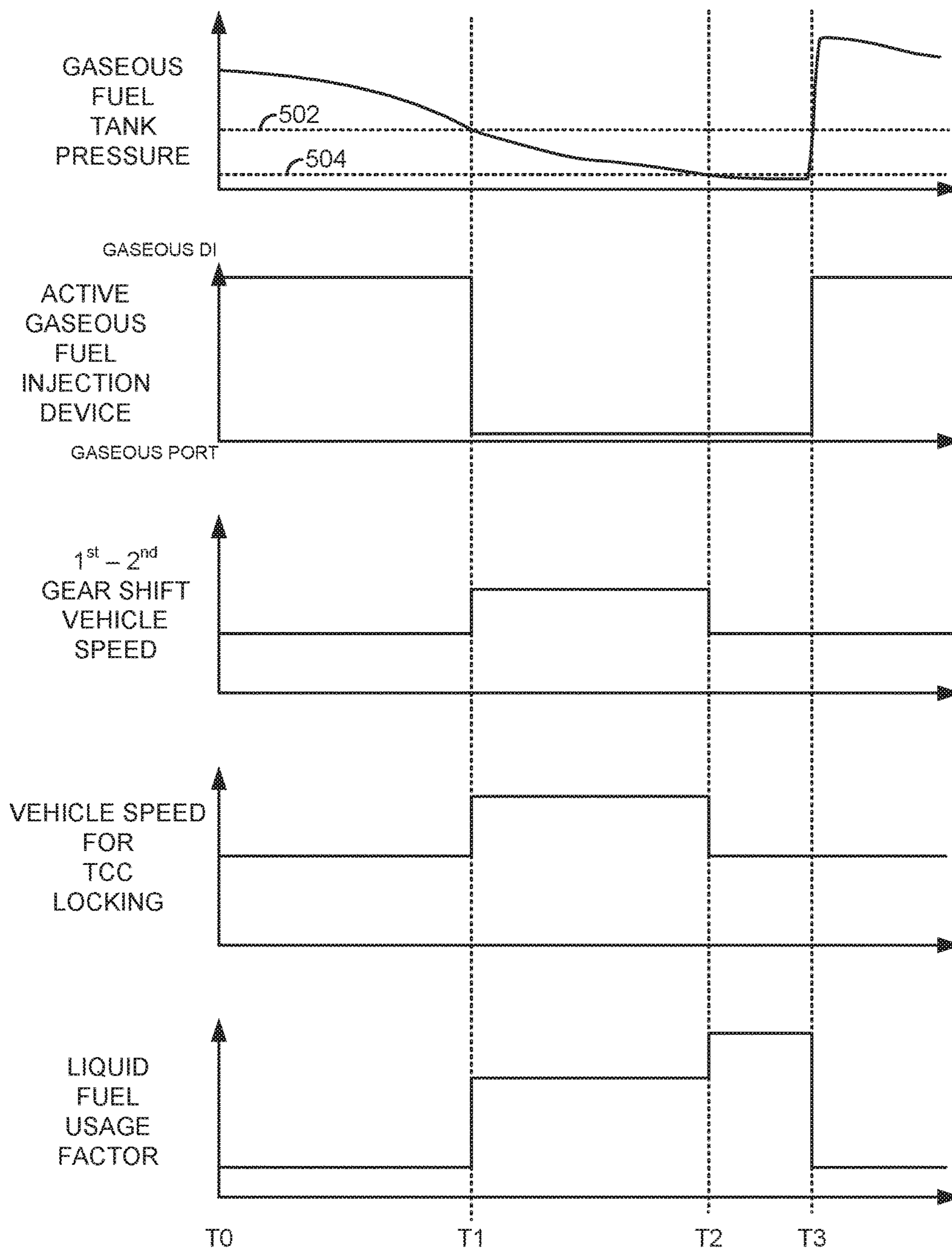


FIG. 5

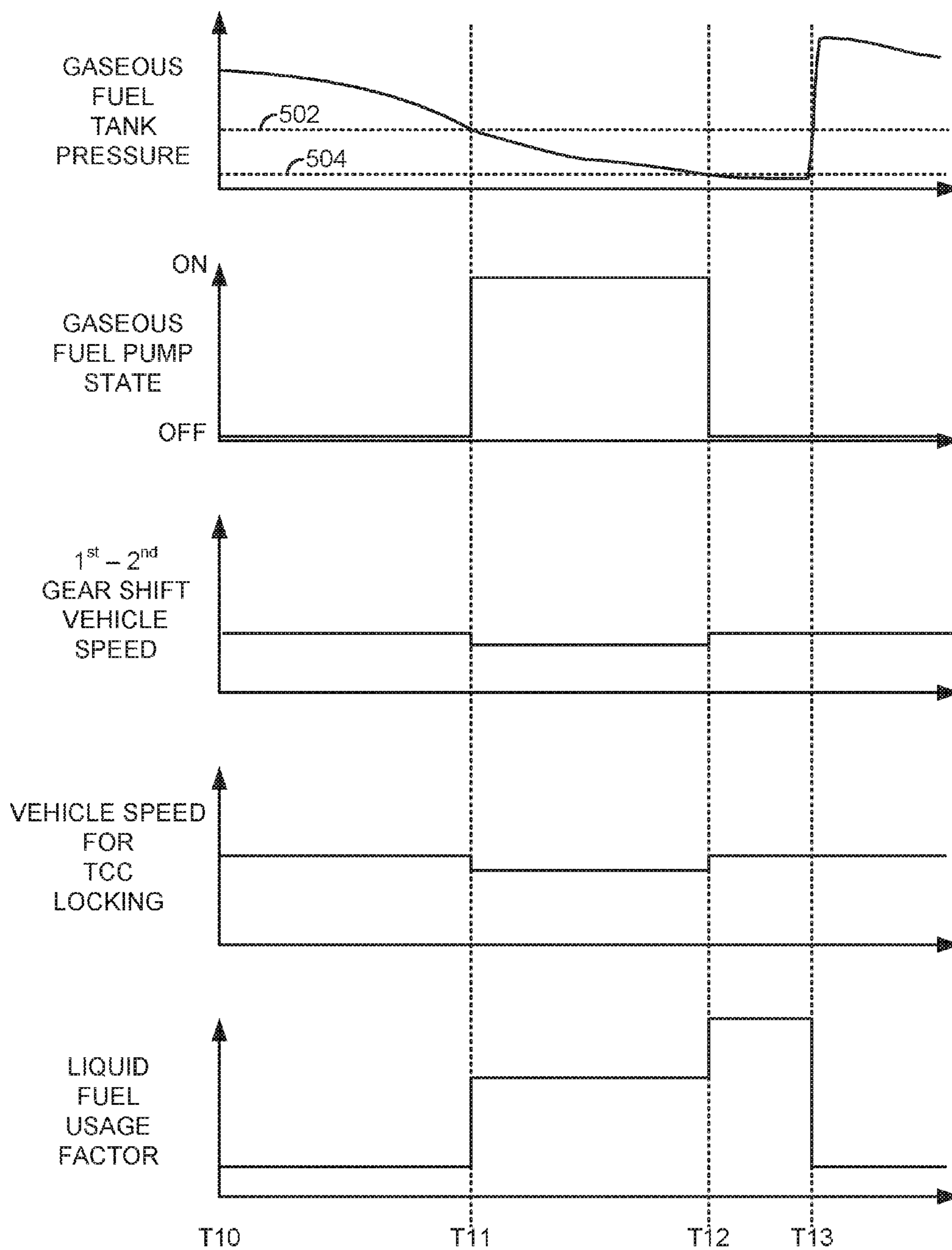


FIG. 6

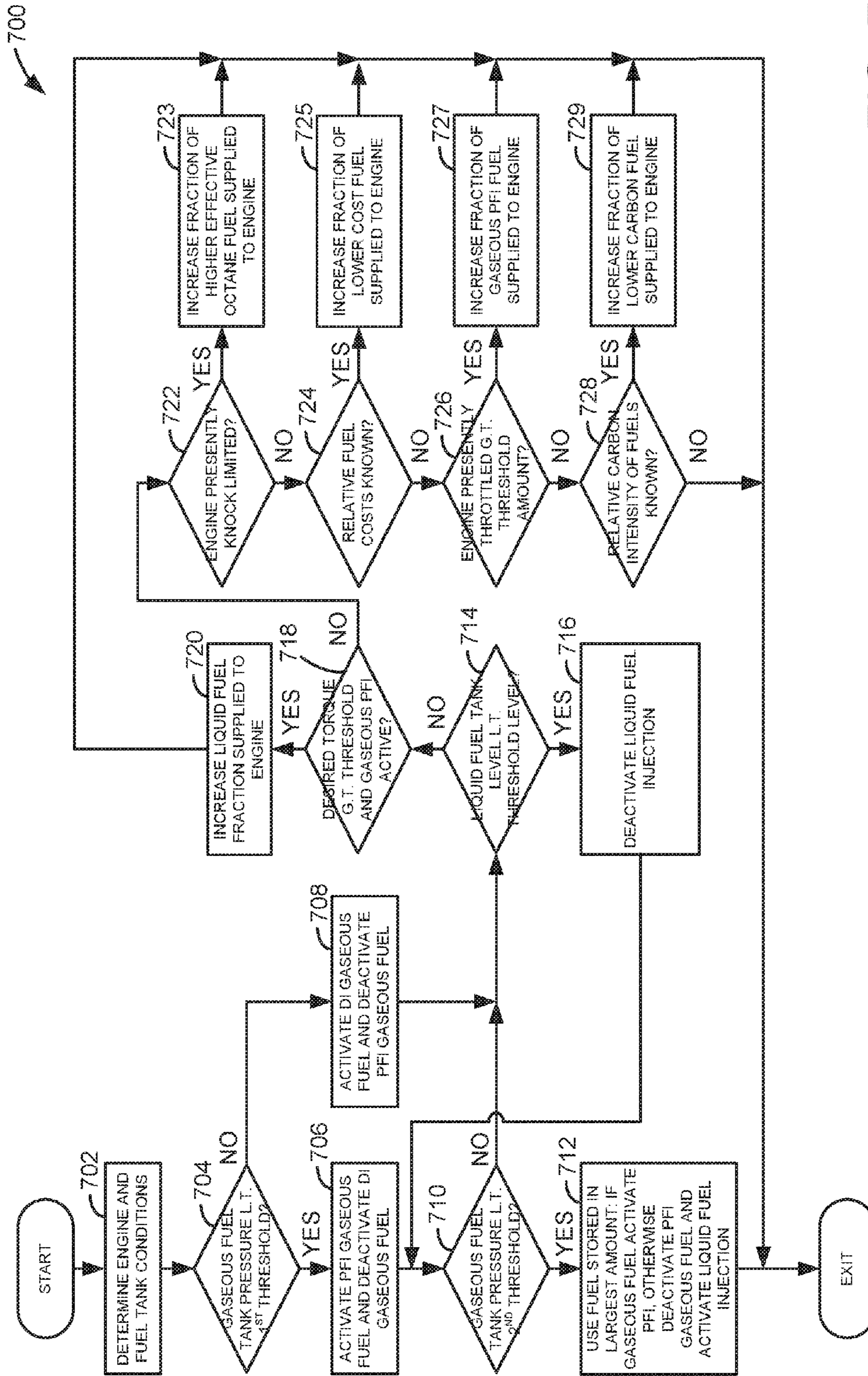


FIG. 7

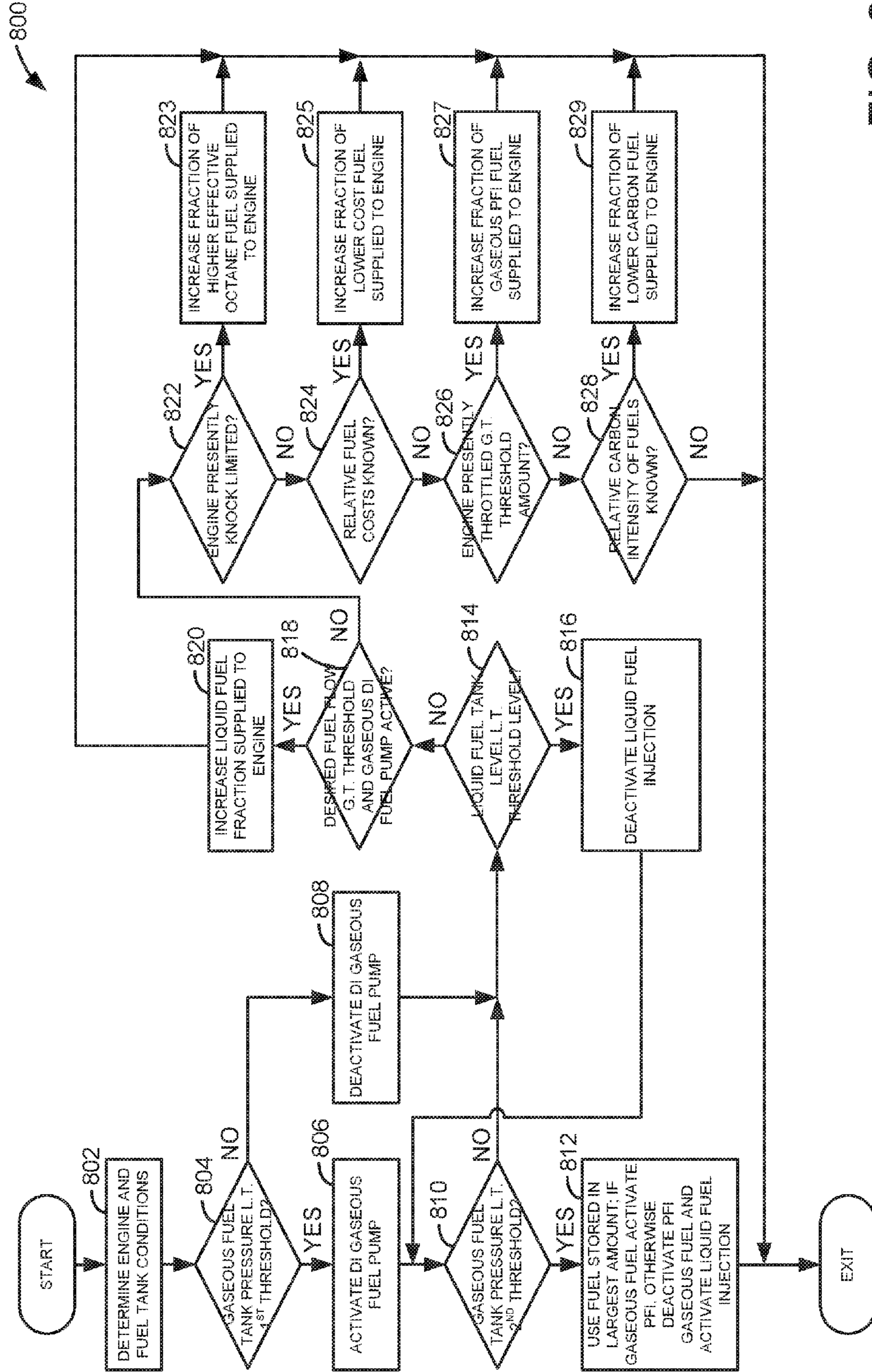


FIG. 8

SYSTEMS AND METHODS FOR IMPROVING TORQUE RESPONSE OF AN ENGINE

CROSS REFERENCE TO RELATED APPLICATIONS

The present application is a continuation of U.S. patent application Ser. No. 14/260,970, entitled "SYSTEMS AND METHODS FOR IMPROVING TORQUE RESPONSE OF AN ENGINE," filed on Apr. 24, 2014, the entire contents of which are hereby incorporated by reference for all purposes.

FIELD

The present description relates to systems and methods for improving torque response of an engine. The methods may be particularly useful for engines that may exhibit a lag in torque production in response to an increase in engine torque demand.

BACKGROUND AND SUMMARY

Engine torque may be increased in response to a change in a requested or desired engine torque. Engine torque may be increased via increasing an amount of air and fuel supplied to the engine, at least up to a condition where the engine lacks capacity to induct additional air into engine cylinders. The amount of fuel supplied to the engine may be increased as the amount of air inducted to the engine increases such that the engine operates near stoichiometric conditions. However, there may be conditions where the amount of air inducted to engine cylinders is less than desired. For example, if the engine is supplied gaseous fuel, the gaseous fuel may displace some air in the cylinder, thereby limiting the amount of torque the engine may produce. Consequently, a desired engine torque response may not be provided.

The inventor herein has recognized the above-mentioned disadvantages and has developed an engine operating method, comprising: directly injecting a gaseous fuel to an engine in response to a pressure of gaseous fuel stored in a fuel tank exceeding a threshold level; and increasing an idle speed of the engine and injecting the gaseous fuel to an engine air intake in response to the amount of gaseous fuel stored in the fuel tank being less than the threshold level.

By increasing idle speed of an engine in response to a pressure of fuel stored in a fuel tank, it may be possible to improve an engine's torque response. For example, if a gaseous fuel is directly injected to engine cylinders, the engine may respond quickly to an increase in desired engine torque. However, if the engine transitions to injecting the gaseous fuel via engine intake injection (e.g., injecting fuel to an engine intake manifold or intake ports) when higher pressure gaseous fuel is not available from the gaseous fuel storage tank, engine output torque at engine idle may not respond as quickly as is desired. The engine torque response at lower engine speeds may be dampened since the engine may generate less torque at lower engine speeds because a portion of cylinder volume may be displaced by fuel. However, if engine idle speed is increased, engine power output may increase more rapidly in a short period of time since more combustion events occur for each second of engine operation. Thus, performance of an engine having engine intake injection may be improved so that vehicle acceleration may be more consistent between direct injection and engine intake injection modes.

The present description may provide several advantages. In particular, the approach may reduce improve engine torque response. Further, the approach may make changes between fuel injection modes less noticeable to a driver. Further still, the approach may extend a vehicle's travel range while operating on a gaseous fuel.

The above advantages and other advantages, and features of the present description will be readily apparent from the following Detailed Description when taken alone or in connection with the accompanying drawings.

It should be understood that the summary above is provided to introduce in simplified form a selection of concepts that are further described in the detailed description. It is not meant to identify key or essential features of the claimed subject matter, the scope of which is defined uniquely by the claims that follow the detailed description. Furthermore, the claimed subject matter is not limited to implementations that solve any disadvantages noted above or in any part of this disclosure.

BRIEF DESCRIPTION OF THE DRAWINGS

The advantages described herein will be more fully understood by reading an example of an embodiment, referred to herein as the Detailed Description, when taken alone or with reference to the drawings, where:

FIG. 1 is a schematic diagram of an engine;

FIG. 2 is shows an example vehicle driveline configuration;

FIGS. 3 and 4 show schematics of example fuel systems;

FIGS. 5 and 6 show example vehicle operating sequences; and

FIGS. 7 and 8 show example methods for operating a vehicle.

DETAILED DESCRIPTION

The present description is related to operating a vehicle that includes an internal combustion engine. The engine may be configured as shown in FIG. 1. The engine may be mechanically coupled to other vehicle components to form a driveline as shown in FIG. 2. The engine may be supplied gaseous and/or liquid fuel via one of the systems shown in FIGS. 3 and 4. The vehicle may operate according to the simulated operating sequences shown in FIGS. 5 and 6. The methods shown in FIGS. 7 and 8 may be included in the systems of FIGS. 1-4 and may provide the sequences in FIGS. 5 and 6.

Referring to FIG. 1, internal combustion engine 10, comprising a plurality of cylinders, one cylinder of which is shown in FIG. 1, is controlled by electronic engine controller 12. Engine 10 includes combustion chamber 30 and cylinder walls 32 with piston 36 positioned therein and connected to crankshaft 40. Flywheel 97 and ring gear 99 are coupled to crankshaft 40. Starter 96 includes pinion shaft 98 and pinion gear 95. Pinion shaft 98 may selectively advance pinion gear 95 to engage ring gear 99. Starter 96 may be directly mounted to the front of the engine or the rear of the engine. In some examples, starter 96 may selectively supply torque to crankshaft 40 via a belt or chain. In one example, starter 96 is in a base state when not engaged to the engine crankshaft. Combustion chamber 30 is shown communicating with intake manifold 44 and exhaust manifold 48 via respective intake valve 52 and exhaust valve 54. Each intake and exhaust valve may be operated by an intake cam 51 and an exhaust cam 53. The position of intake cam 51 may be

determined by intake cam sensor **55**. The position of exhaust cam **53** may be determined by exhaust cam sensor **57**.

Liquid fuel injector **66** is shown positioned to inject fuel directly into cylinder **30**, which is known to those skilled in the art as direct injection. Alternatively, liquid fuel may be injected to an intake port, which is known to those skilled in the art as port injection. Gaseous direct fuel injector **68** supplies gaseous fuel directly into combustion chamber **30**. Engine intake injector (e.g., manifold central gaseous injector) **67a** injects gaseous into intake manifold **44**. Alternatively, engine intake injector (e.g., port gaseous injector) **67b** injects gaseous fuel into cylinder intake port **49**. Injector **67b** is shown as an extended version of injector **67a**, but in some examples injector **67b** may directly enter cylinder intake port **49** without entering intake manifold **44**. Fuel injectors **66**, **67a**, **67b**, and **68** deliver liquid or gaseous fuel in proportion to pulse widths from controller **12**. Fuel is delivered to fuel injector **66** by a fuel system (not shown) including a fuel tank, fuel pump, and fuel rail (not shown). Fuel systems supplying fuel to injectors **66**, **67a**, **67b**, and **68** are shown in FIGS. **3** and **4**.

Intake manifold **44** is shown communicating with optional electronic throttle **62** which adjusts a position of throttle plate **64** to control air flow from air intake **42** to intake manifold **44**. In some examples, throttle **62** and throttle plate **64** may be positioned between intake valve **52** and intake manifold **44** such that throttle **62** is a port throttle.

Distributorless ignition system **88** provides an ignition spark to combustion chamber **30** via spark plug **92** in response to controller **12**. Universal Exhaust Gas Oxygen (UEGO) sensor **126** is shown coupled to exhaust manifold **48** upstream of catalytic converter **70**. Alternatively, a two-state exhaust gas oxygen sensor may be substituted for UEGO sensor **126**.

Converter **70** can include multiple catalyst bricks, in one example. In another example, multiple emission control devices, each with multiple bricks, can be used. Converter **70** can be a three-way type catalyst in one example.

Controller **12** is shown in FIG. **1** as a conventional microcomputer including: microprocessor unit **102**, input/output ports **104**, read-only memory **106**, random access memory **108**, keep alive memory **110**, and a conventional data bus. Controller **12** is shown receiving various signals from sensors coupled to engine **10**, in addition to those signals previously discussed, including: engine coolant temperature (ECT) from temperature sensor **112** coupled to cooling sleeve **114**; a measurement of engine manifold pressure (MAP) from pressure sensor **122** coupled to intake manifold **44**; an engine position sensor from a Hall effect sensor **118** sensing crankshaft **40** position; a measurement of air mass entering the engine from sensor **120**; and a measurement of throttle position from sensor **58**. Barometric pressure may also be sensed (sensor not shown) for processing by controller **12**. In a preferred aspect of the present description, engine position sensor **118** produces a predetermined number of equally spaced pulses every revolution of the crankshaft from which engine speed (RPM) can be determined.

During operation, each cylinder within engine **10** typically undergoes a four stroke cycle: the cycle includes the intake stroke, compression stroke, expansion stroke, and exhaust stroke. During the intake stroke, generally, the exhaust valve **54** closes and intake valve **52** opens. Air is introduced into combustion chamber **30** via intake manifold **44**, and piston **36** moves to the bottom of the cylinder so as to increase the volume within combustion chamber **30**. The position at which piston **36** is near the bottom of the cylinder

and at the end of its stroke (e.g. when combustion chamber **30** is at its largest volume) is typically referred to by those of skill in the art as bottom dead center (BDC). During the compression stroke, intake valve **52** and exhaust valve **54** are closed. Piston **36** moves toward the cylinder head so as to compress the air within combustion chamber **30**. The point at which piston **36** is at the end of its stroke and closest to the cylinder head (e.g. when combustion chamber **30** is at its smallest volume) is typically referred to by those of skill in the art as top dead center (TDC). In a process hereinafter referred to as injection, fuel is introduced into the combustion chamber. In a process hereinafter referred to as ignition, the injected fuel is ignited by known ignition means such as spark plug **92**, resulting in combustion. During the expansion stroke, the expanding gases push piston **36** back to BDC. Crankshaft **40** converts piston movement into a rotational torque of the rotary shaft. Finally, during the exhaust stroke, the exhaust valve **54** opens to release the combusted air-fuel mixture to exhaust manifold **48** and the piston returns to TDC. Note that the above is shown merely as an example, and that intake and exhaust valve opening and/or closing timings may vary, such as to provide positive or negative valve overlap, late intake valve closing, or various other examples.

FIG. **2** is a block diagram of a vehicle driveline **200**. Driveline **200** may be powered by engine **10** in vehicle **290**. Engine **10** may be started with an engine starting system shown in FIG. **1**. Further, engine **10** may generate or adjust torque via torque actuator **204**, such as a fuel injector, throttle, etc.

An engine output torque may be transmitted to torque converter **206** via shaft **275** and impeller **285**. Torque converter **206** includes a turbine **286** to output torque to transmission input shaft **270**. Input shaft **270** mechanically couples torque converter **206** to automatic transmission **208**. Torque converter **206** also includes a torque converter bypass lock-up clutch **212** (TCC). Torque is directly transferred from impeller **285** to turbine **286** when TCC is locked. TCC is electrically operated by controller **12**. Alternatively, TCC may be hydraulically locked. In one example, the torque converter may be referred to as a component of the transmission.

When torque converter lock-up clutch **212** is fully disengaged, torque converter **206** transmits engine torque to automatic transmission **208** via fluid transfer between the torque converter turbine **286** and torque converter impeller **285**, thereby enabling torque multiplication. In contrast, when torque converter lock-up clutch **212** is fully engaged, the engine output torque is directly transferred via the torque converter clutch to input shaft **270** of transmission **208**. Alternatively, the torque converter lock-up clutch **212** may be partially engaged, thereby enabling the amount of torque directly relayed to automatic transmission **208** to be adjusted. The controller **12** may be configured to adjust the amount of torque transmitted by torque converter **212** by adjusting the torque converter lock-up clutch in response to various engine operating conditions, or based on a driver-based engine operation request.

Automatic transmission **208** includes gear clutches (e.g., gears 1-6) **211** and forward clutch **210**. The gear clutches **211** and the forward clutch **210** may be selectively engaged to propel a vehicle. Torque output from the automatic transmission **208** may in turn be relayed to wheels **216** to propel the vehicle via output shaft **260**. Specifically, automatic transmission **208** may transfer an input driving torque at the

input shaft **270** responsive to a vehicle traveling condition before transmitting an output driving torque to the wheels **216**.

Further, a frictional force may be applied to wheels **216** by engaging wheel brakes **218**. In one example, wheel brakes **218** may be engaged in response to the driver pressing his foot on a brake pedal (not shown). In other examples, controller **12** or a controller linked to controller **12** may apply engage wheel brakes. In a similar way, a frictional force may be reduced to wheels **216** by disengaging wheel brakes **218** in response to the driver releasing his foot from a brake pedal. Further, vehicle brakes may apply a frictional force to wheels **216** via controller **12** as part of an automated engine stopping procedure.

Controller **12** may be configured to receive inputs from engine **10**, as shown in more detail in FIG. **1**, and accordingly control a torque output of the engine and/or operation of the torque converter, transmission, clutches, and/or brakes. As one example, an engine torque output may be controlled by adjusting a combination of spark timing, fuel pulse width, fuel pulse timing, and/or air charge, by controlling throttle opening and/or valve timing, valve lift and boost for turbo- or super-charged engines.

Referring now to FIG. **3**, an example fuel system for supplying fuel to engine **10** of FIG. **1** is shown. The fuel system of FIG. **3** may be operated according to the method of FIG. **7**. Further, the fuel system of FIG. **3** may be part of a system providing the operating sequence shown in FIG. **5**.

Fuel system **300** includes a liquid fuel tank **301** and a gaseous fuel tank **354**. The liquid fuel tank **301** may store gasoline, alcohol, or a mixture of gasoline and alcohol. The gaseous fuel tank **354** may store compressed natural gas (CNG), liquefied petroleum gas (LPG which changes state to a gas upon injection), hydrogen, or other gaseous fuel.

Liquid fuel tank **301** includes a fuel level sensor **311** and a fuel pump **302**. Fuel pump **302** may be electrically driven via a command from controller **12**. Fuel pump **302** may be a low pressure fuel pump and it supplies fuel to high pressure fuel pump **306** via conduit **304**. High pressure fuel pump **306** supplies fuel to fuel rail **308** and it distributes fuel to fuel injector **66**. High pressure fuel pump **306** may be driven by engine **10**. Pressure at fuel rail **308** may be monitored via pressure sensor **325**. Controller **12** may adjust an amount of fuel pumped by high pressure fuel pump **306** in response to output of pressure sensor **325**.

Gaseous fuel tank **354** includes a pressure sensor **358** for judging an amount of fuel stored in fuel tank **354**. Regulator **352** adjusts fuel pressure from fuel tank **354** to a constant pressure when fuel pressure in fuel tank **354** is greater than a threshold pressure. Regulator **352** directs gaseous fuel to three-way valve **356**. Three-way valve **356** directs gaseous fuel to direct fuel injector **68** or port fuel injector **67a** (or alternatively **67b**) based on output from controller **12**.

Thus, the system of FIGS. **1-3** provides for a vehicle system, comprising: an engine; a gaseous fuel tank; and a controller including non-transitory instructions for increasing engine idle speed and adjusting a transmission shifting schedule in response to the amount of gaseous fuel stored in the gaseous fuel tank being less than the threshold level. The vehicle system further comprises non-transitory instructions for adjusting a torque converter lock-up schedule in response to the amount of gaseous fuel stored in the gaseous fuel tank being less than the threshold level. The vehicle system further comprises a step ratio transmission, and where adjusting the transmission shifting schedule increases a vehicle speed at which the step ratio transmission is shifted. The vehicle system further comprises adjusting a

torque at which the step ratio transmission is shifted. The vehicle system includes where the torque at which the step ratio transmission is shifted is reduced in response to the amount of gaseous fuel stored in the gaseous fuel tank being less than the threshold level. The vehicle system further comprises ceasing direct injection of a gaseous fuel to the engine and starting port injection of the gaseous fuel in response to the amount of gaseous fuel stored in the gaseous fuel tank being less than the threshold level.

Referring now to FIG. **4**, a second example fuel system for supplying fuel to engine **10** of FIG. **1** is shown. The fuel system of FIG. **4** may be operated according to the method of FIG. **8**. Further, the fuel system of FIG. **4** may be part of a system providing the operating sequence shown in FIG. **6**. The fuel system of FIG. **4** includes many of the same components as the fuel system of FIG. **3**. Like components between FIG. **3** and FIG. **4** are included in FIG. **4** using the same numeric identifiers. Therefore, for the sake of brevity, the description of like components is omitted from the description of FIG. **4**. However, like components operate as described in FIG. **3**.

Fuel system **400** includes a gaseous fuel pump **465** which is supplied gaseous fuel from gaseous fuel tank **354**. Gaseous fuel pump **465** may be driven by engine **10** or it may be electrically driven. Gaseous fuel pump **465** may be selectively activated and deactivated by controller **12** in response to a fuel amount stored in gaseous fuel tank **354** and/or an amount of liquid fuel stored in fuel tank **301**. Gaseous fuel pump **465** may supply pressurized gaseous fuel to accumulator **460**, or alternatively three-way valve **456**. The output pressure of accumulator **460** may be adjusted to a desired pressure via regulator **462**, or the pressure of accumulator **460** may be adjusted by modulating speed or displacement of gaseous fuel pump **465**. The operating state (e.g., on/off or pumping capacity) of gaseous fuel pump **465** may be adjusted in response to pressure sensed in accumulator **460** via pressure sensor **471**.

In one example, controller **12** supplies gaseous fuel to engine **10** from gaseous fuel tank **354** via pressure regulator **352**, three-way valve **456**, and direct injector **68** when fuel pressure in fuel tank **354** is greater than a threshold amount. Gaseous fuel pump **465** is deactivated when fuel pressure or the amount of gaseous fuel stored in fuel tank **354** is greater than the threshold amount. Fuel is not supplied to engine **10** via regulator **462** when fuel is supplied to engine **10** via regulator **352**. If the amount of fuel stored in gaseous fuel tank is less than the threshold amount, gaseous fuel pump **465** is activated and gaseous fuel is supplied to engine **10** from gaseous fuel tank **354** via gaseous fuel pump **465**, accumulator **460**, pressure regulator **462**, three-way valve **456**, and direct fuel injector **68**. Fuel is not supplied to engine **10** via regulator **352** when gaseous fuel pump **465** is activated.

Thus, the system of FIGS. **1, 2, and 4** provides for a vehicle system, comprising: an engine; a gaseous fuel tank; a gaseous fuel pump in pneumatic communication with the gaseous fuel tank; and a controller including non-transitory instructions for supplying gaseous fuel to the engine and activating the gaseous fuel pump only when a pressure in the gaseous fuel tank is less than a threshold pressure. The vehicle system further comprises non-transitory instructions for adjusting a torque converter lock-up schedule in response to the pressure in the gaseous fuel tank is less than the threshold pressure. The vehicle system further comprises a fixed gear ratio transmission, and non-transitory instructions for adjusting a transmission shifting schedule in response to the pressure in the gaseous fuel tank being less

than the threshold pressure. The vehicle system may further comprise an accumulator positioned downstream of the gaseous fuel pump. The vehicle system further comprises a pressure regulator positioned downstream of the accumulator. The vehicle system may further comprise a direct gaseous fuel injector coupled to the engine.

Referring now to FIG. 5, an example vehicle operating sequence according to the method of FIG. 7 is shown. The sequence of FIG. 5 may be provided via the method of FIG. 7 and the system as shown in FIGS. 1, 2, and 3.

The first plot from the top of FIG. 5 is a plot of gaseous fuel pressure in the gaseous fuel tank versus time. The Y axis represents gaseous fuel pressure and the pressure increases in the direction of the Y axis arrow. The X axis represents time and time increases from the left side of the figure to the right side of the figure. Horizontal line 502 represents a first threshold level below which the engine is operated with port or air intake gaseous fuel injectors. The engine operates with direct fuel injectors when the gaseous fuel storage amount is greater than threshold 502. Horizontal line 504 represents a second threshold level below which the engine is operated solely with liquid fuel unless the amount of stored liquid fuel is less than a threshold level.

The second plot from the top of FIG. 5 is a plot of state of the active gaseous fuel injection device versus time. The active gaseous fuel injection device may be a port injector for injecting gaseous fuel into the intake manifold or intake ports, or alternatively, the active gaseous fuel injection device may be a direct fuel injector for injecting fuel directly into an engine cylinder. The active gaseous fuel injection device is the direct fuel injector when the trace is near the Y axis arrow. The active gaseous fuel injection device is the port fuel injector when the trace is near the X axis. The X axis represents time and time increases from the left side of the figure to the right side of the figure.

The third plot from the top of FIG. 5 is an example plot of a scheduled vehicle speed at which first gear to second gear upshift occurs versus time. The Y axis represents scheduled vehicle speed for first gear to second gear upshifts and vehicle speed increases in the direction of the Y axis arrow. The X axis represents time and time increases from the left side of the figure to the right side of the figure. It is to be understood that gear shifting strategy is also a function of engine torque (or a surrogate for torque such as intake manifold pressure or accelerator pedal position). This plot represents upshift speed for a typical engine torque, and the upshift speed for other torques would behave in a similar manner. Likewise, the vehicle speeds for upshifting from second gear to third gear would behave in a similar manner, and so on.

The fourth plot from the top of FIG. 5 is a plot of a scheduled vehicle speed at which the torque converter clutch is locked for a particular selected gear versus time. The Y axis represents scheduled vehicle speed at which the torque converter clutch is locked for a particular selected gear and vehicle speed increases in the direction of the Y axis arrow. The X axis represents time and time increases from the left side of the figure to the right side of the figure. It is understood that torque converter clutch lockup may be scheduled as a function of engine speed rather than vehicle speed, with similar results.

The fifth plot from the top of FIG. 5 is a plot of liquid fuel usage factor versus time. The Y axis represents liquid fuel usage factor and the liquid fuel usage factor increases in the direction of the Y axis arrow. The X axis represents time and time increases from the left side of the figure to the right side of the figure.

At time T0, the pressure of gaseous fuel stored in the gaseous fuel tank is at a higher level. Direct gaseous fuel injection is activated in response to the pressure of gaseous fuel stored in the gaseous fuel tank. The vehicle speed at which a first gear to second gear upshift is scheduled is at a lower vehicle speed so that upshifting occurs earlier in time for a vehicle that is accelerating. The vehicle speed at which the torque converter clutch is locked is scheduled is a lower vehicle speed so that torque converter lockup occurs earlier in time for a vehicle that is accelerating. The liquid fuel usage factor is at a lower level and it indicates that only a small fraction of a total fuel amount supplied to the engine during an engine cycle is comprised of liquid fuel.

Between time T0 and time T1, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced in response to the engine consuming fuel. The direct gaseous fuel injectors remain active, the first gear to second gear upshift schedule is unchanged, the vehicle speed at which the torque converter clutch is locked remains the same, and the liquid fuel usage factor remains the same.

At time T1, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced to less than threshold level 502 as the engine consumes gaseous fuel. Therefore, the active gaseous fuel injector is switched from direct fuel injection to port or intake fuel injection. Further, the vehicle speed at which a first gear to second gear upshift occurs is increased. Likewise, the vehicle speed at which the torque converter clutch is locked is increased in response to the pressure of gaseous fuel stored in the gaseous fuel tank. Additionally, the liquid fuel usage factor is increased so that a greater fraction of a total amount of fuel supplied to the engine during an engine cycle is comprised of liquid fuel.

Between time T1 and time T2, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced further in response to the engine consuming fuel. The port gaseous fuel injectors remain active, the first gear to second gear upshift schedule is unchanged, the vehicle speed at which the torque converter clutch is locked remains the same, and the liquid fuel usage factor remains the same.

At time T2, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced to less than threshold level 504. Consequently, the active gaseous fuel injectors remain the port fuel injectors, but gaseous fuel injection is stopped since there is sufficient liquid fuel available (not shown) to operate the engine. Further, the vehicle speed at which a first gear to second gear upshift occurs is decreased to its former level. Likewise, the vehicle speed at which the torque converter clutch is locked is decreased to its former level in response to switching over to solely injecting liquid fuel. The liquid fuel usage factor is increased so that the engine is supplied solely liquid fuel.

At time T3, the gaseous fuel storage tank is refilled. Consequently, the active gaseous fuel injector is switched to direct injection. The direct gaseous fuel injectors are activated, port gaseous fuel injectors are deactivated, and the liquid fuel usage factor is reduced so that the fraction of liquid fuel supplied to the engine is reduced in response to the gaseous fuel tank being filled to a level greater than 502. The vehicle speed at which a first gear to second gear upshift occurs is at a same level as when only liquid fuel is injected to the engine. Likewise, the vehicle speed at which the torque converter clutch is locked is maintained at the same level as when only liquid fuel is injected to the engine.

Thus, the fuel injector type, gear shift schedule, and torque converter lock schedule are adjusted in response to pressure of fuel stored in the gaseous fuel tank. By changing from direct gaseous fuel injection to port gaseous fuel

injection, a greater amount of gaseous fuel stored in the gaseous fuel tank may be utilized since the gaseous fuel may be port injected at a lower fuel pressure; however, the amount of engine torque and power available may be reduced since a portion of cylinder volume is occupied by gaseous fuel that was drawn into the cylinder from the engine air intake. Additionally, adjusting the transmission shift schedule and the torque converter clutch lockup schedule allows the engine to produce additional power when gaseous port fuel injection used, so that there is less possibility of the engine producing less than desired power after a shift or after the torque converter is locked.

Referring now to FIG. 6, an example vehicle operating sequence according to the method of FIG. 8 is shown. The sequence of FIG. 6 may be provided via the method of FIG. 8 and the system as shown in FIGS. 1, 2, and 4. FIG. 6 includes plots similar to the plots of FIG. 5, except that the second plot of FIG. 6 is a plot of gaseous fuel pump state versus time instead of fuel injection type. Therefore, for the sake of brevity, the description of plots of FIG. 5, similar to the plots in FIG. 6, are not repeated. However, the description of variables from FIG. 5 applies to the same variables in FIG. 6.

The second plot from the top of FIG. 6 is a plot of gaseous fuel pump state versus time. The gaseous fuel pump is activated when the trace is at a higher level. The gaseous fuel pump is deactivated when the trace is at a lower level.

At time T10, the pressure of gaseous fuel stored in the gaseous fuel tank is at a higher level. The gaseous fuel pump is not activated in response to the pressure of gaseous fuel stored in the gaseous fuel tank. The vehicle speed at which a first gear to second gear upshift is scheduled is at standard vehicle speed. The vehicle speed at which the torque converter clutch is locked is scheduled at a standard vehicle speed. The liquid fuel usage factor is at a lower level and it indicates that only a small fraction of a total fuel amount supplied to the engine during an engine cycle is comprised of liquid fuel.

Between time T10 and time T11, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced in response to the engine consuming fuel. The gaseous fuel pump remains active, the first gear to second gear upshift schedule is unchanged, the vehicle speed at which the torque converter clutch is locked remains the same, and the liquid fuel usage factor remains the same.

At time T11, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced to less than threshold level 502. Therefore, the active gaseous fuel pump is activated so that gaseous fuel is supplied to direct injectors at sufficient pressure to enter the cylinder. The vehicle speed at which a first gear to second gear upshift occurs is decreased since engine speed may be limited by the gaseous fuel pump flow capacity. Likewise, the vehicle speed at which the torque converter clutch is locked is reduced since engine speed may be limited while the gaseous fuel pump is activated. Additionally, the liquid fuel usage factor is increased so that a greater fraction of a total amount of fuel supplied to the engine during an engine cycle is comprised of liquid fuel.

Between time T11 and time T12, the pressure of gaseous fuel stored in the gaseous fuel tank is reduced further in response to the engine consuming fuel. The gaseous fuel pump remains active, the first gear to second gear upshift schedule is unchanged, the vehicle speed at which the torque converter clutch is locked remains the same, and the liquid fuel usage factor remains the same.

At time T12, the amount of gaseous fuel stored in the gaseous fuel tank is reduced to less than threshold level 504.

As a result, the gaseous fuel pump is deactivated (not shown) and the liquid fuel fraction is increased so that the engine is supplied solely liquid fuel. Additionally, the vehicle speed at which a first gear to second gear upshift occurs is increased to its former level when the gaseous fuel pump was not active. Likewise, the vehicle speed at which the torque converter clutch is locked is increased to its former level in response to switching over to solely injecting liquid fuel.

At time T13, the gaseous fuel storage tank is refilled. Consequently, the direct gaseous fuel injectors are activated and the liquid fuel usage factor is reduced so that the fraction of liquid fuel supplied to the engine is reduced. In this way, gaseous fuel injection and liquid fuel injection are adjusted in response to the gaseous fuel tank being filled to a pressure greater than 502. The vehicle speed at which a first gear to second gear upshift occurs is at a same level as when the gaseous fuel tank was filled to a pressure greater than 502. Likewise, the vehicle speed at which the torque converter clutch is locked is maintained at the same level as when the gaseous fuel tank was filled to a pressure greater than 502.

Thus, the gaseous fuel pump may be activated and deactivated in response to a pressure of gaseous fuel stored in a fuel tank. By operating the gaseous fuel pump, gaseous fuel at a lower pressure can be converted to high pressure gaseous fuel suitable for direct injection. However, in some example, the gaseous fuel flow rate may be limited such that it may be desirable to adjust the gear shift schedules and torque converter lock schedule to shift and lock at lower vehicle speeds.

Referring now to FIG. 7, a method for operating a vehicle is shown. The method may be incorporated into the system of FIGS. 1-3 as executable instructions stored in non-transitory memory. The method of FIG. 7 may provide the operating sequence shown in FIG. 5.

At 702, method 700 determines engine and fuel tank conditions. Engine conditions may include but are not limited to engine speed, driver demand torque, present type of fuel injected, and engine temperature. Fuel tank conditions may include but are not limited to fuel level or amount of fuel stored in a fuel tank, fuel temperature, fuel pressure, fuel pump state (e.g., on or off), and fuel delivery path (e.g., port injection/direct injection). Method 700 proceeds to 704 after engine and fuel tank conditions are determined.

At 704, method 700 judges whether or not gaseous fuel tank pressure is less than (L.T.) a first threshold pressure. The first threshold pressure may be indicative of a fuel pressure for overcoming cylinder pressure to allow fuel into the cylinder while directly injecting fuel to a cylinder. Further, the first threshold pressure may vary with engine operating conditions (e.g., fuel injection timing, engine speed and demand torque). If method 700 judges that the gaseous fuel tank pressure is less than the first threshold pressure, the answer is yes and method 700 proceeds to 706. Otherwise, the answer is no and method 700 proceeds to 708.

At 706, method 700 deactivates direct gaseous fuel injectors and activates port or intake manifold gaseous fuel injectors. The port or intake manifold gaseous injectors are activated so that gaseous fuel may be supplied to the engine cylinders even when pressure in the gaseous fuel tank is too low to support direct injection. Consequently, the vehicle's driving range may be extended as compared to if the engine were only operated with direct gaseous fuel injection. Method 700 proceeds to 710 after port gaseous fuel injectors are activated.

Additionally, when method **700** switches to gaseous port or intake manifold injection from gaseous direct fuel injection, method **700** adjusts engine idle speed, transmission gear shift schedules, and torque converter lockup scheduling. In one example, the engine idle speed is increased when gaseous port fuel injection is activated and gaseous direct fuel injection is deactivated. The amount of increase in engine idle speed may be based on the type of gaseous fuel being injected. For example, if method **700** changes from gaseous fuel direct injection to gaseous fuel port injection, idle speed may be increased by 200 RPM. By increasing idle speed, an amount of time it takes to increase engine torque from a time an increase in torque is requested until the engine produces the requested torque may be reduced. The higher idle speed increases the number of combustion events per second. Therefore, even if a portion of cylinder volume is displaced by gaseous fuel rather than air, engine torque can be increased quickly since torque is provided each combustion event and since a higher idle speed yields more combustion events per second. Additionally, engine idle speed may be increased further in response to operating the engine at lower barometric pressure. For example, if the engine operates at increasing altitude, the engine idle speed may be increased further to improve the engine's torque response while operating the engine with port injected gaseous fuel.

The transmission gear shifting schedule is also adjusted based on the method of gaseous fuel injection. For example, if the engine switches from direct gaseous fuel injection to gaseous port injection, the vehicle speed at which each transmission gear upshift occurs is increased. Thus, a first gear to second gear upshift at a first engine torque demand may be increased from a vehicle speed of 16 Kilometers per hour (Kph) to a vehicle speed of 22 Kph so that engine speed may be maintained at a higher speed after the transmission gear shift. Operating the engine at a higher speed after a shift may allow the engine to produce more power and provide improved performance as compared to if the transmission were shifted at lower vehicle speeds during a period when gaseous fuel is port injected to the engine. The upshifting schedule for higher transmission gears may be adjusted similarly. Downshifting schedules may also be adjusted so that the transmission downshifts at a higher vehicle speed when operated with gaseous fuel port injection as compared to when the engine and vehicle are operated at similar conditions with gaseous fuel direct injection. For example, a third gear to second gear downshift may be scheduled at vehicle speed of 48 Kph when the engine is operated with gaseous port fuel injection and a vehicle speed of 42 Kph when the engine is operated with gaseous direct fuel injection. The downshifting schedule for other transmission gears may be adjusted similarly.

In some examples, the engine or wheel torque where gear upshifts and downshifts occur may also be adjusted similar to the way the vehicle speed upshift gear shifts are adjusted. For example, if the engine transmission is scheduled to upshift from first gear to second gear at 16 Kph and 65 N-m of driver demand or wheel torque when gaseous fuel is directly injected to the engine, the same first gear to second gear upshift may be scheduled at 22 Kph and 80 N-m of driver demand or wheel torque when gaseous fuel is port injected to the engine. Similar shift schedules may be applied to all transmission gears.

The transmission torque converter lockup schedule is also adjusted based on the method of gaseous fuel injection. For example, if the engine switches from direct gaseous fuel injection to gaseous port injection, the vehicle speed (or

engine speed) at which the torque converter lockup occurs is increased. Thus, a torque converter lockup may be scheduled for vehicle speed of 88 Kph when the engine receives port injected gaseous fuel while a transmission is in fourth gear. On the other hand, the torque converter lockup may be scheduled for vehicle speed of 80 Kph when the engine receives directly injected gaseous fuel while the transmission is in fourth gear. Increasing the torque converter lockup speed allows the engine speed to be at a higher speed after the torque converter is locked so that the engine may be operating at a speed where the engine has the capacity to produce more power. Consequently, the engine may provide more power in a shorter amount of time as compared to if the engine were operating at a lower speed and receiving fuel via gaseous port fuel injection. However, in some examples the torque converter clutch may be held open while port gaseous injection is enabled so that torque converter impeller speed and engine speed may be greater than if the torque converter clutch was locked, thereby improving the engine's performance.

Additionally, in some examples, a fraction of liquid fuel supplied to the engine during an engine cycle may be increased in response to driver demand or wheel torque exceeding a threshold torque. By increasing the fraction of liquid fuel when gaseous fuel is being port injected, it may be possible to increase engine torque output even when gaseous fuel may be displacing some air from the engine cylinders.

At **710**, method **700** judges whether or not gaseous fuel tank pressure is less than (L.T.) a second threshold pressure. The second threshold pressure is less than the first threshold pressure. If so, the answer is yes and method **700** proceeds to **712**. Otherwise, the answer is no and method **700** proceeds to **714**.

At **712**, method **700** uses fuel remaining in the gaseous and liquid fuel tanks to extend the vehicle's driving range even when the amount of fuel stored in the gaseous and liquid fuel tanks is low. In one example, method **700** selects a fuel type to operate the engine with based on the fuel tank that has the greatest amount of stored fuel. For example, if the gaseous fuel tank is one quarter full and the liquid fuel tank is one eighth full, method **700** selects gaseous fuel to operate the engine on and injects the gaseous fuel via port gaseous fuel injectors. In one example, method **700** operates the engine with the selected fuel for a predetermined duration (e.g., amount of time or travel distance) and then switches over to the previously unselected fuel. For example, if the gaseous fuel is selected first, the engine operates with the gaseous fuel for ten minutes and then switches over to operating the engine using liquid fuel for ten minutes and then the engine is operated for another ten minutes with gaseous fuel. If the engine loses power while operating with one fuel, the engine switches over to the other fuel and continues to operate on the selected fuel until the vehicle is refueled or is out of fuel. Method **700** proceeds to exit after selecting fuel to operate the engine with.

At **708**, method **700** activates direct gaseous fuel injectors and deactivates port or intake manifold gaseous fuel injectors. The direct gaseous fuel injectors are activated so that gaseous fuel may be supplied to an engine cylinder during the cylinder's compression stroke when intake valves are closed. Consequently, engine power loss due to gaseous fuel injection may be reduced since air in the cylinder is not displaced by gaseous fuel. Method **700** proceeds to **714** after direct gaseous fuel injectors are activated.

Further, when method **700** switches to gaseous direct injection from gaseous port injection, method **700** adjusts

engine idle speed, transmission gear shift schedules, and torque converter lockup scheduling. In particular, the engine idle speed is decreased when gaseous direct injection is activated and gaseous port fuel injection is deactivated. The amount of decrease in engine idle speed may be based on the type of gaseous fuel being injected. For example, if method **700** changes from gaseous fuel port injection to gaseous fuel direct injection, idle speed may be decreased by 200 RPM. By decreasing idle speed, it may be possible to reduce engine fuel consumption at idle conditions.

The transmission gear shifting schedule is also adjusted based on the method of gaseous fuel injection. Specifically, if the engine switches from port gaseous fuel injection to gaseous direct injection, the vehicle speed at which each transmission gear upshift occurs is decreased. Thus, a first gear to second gear upshift at a first engine torque demand may be decreased from a vehicle speed of 22 Kilometers per hour (Kph) to a vehicle speed of 16 Kph so that engine speed may be maintained at a lower speed after the transmission gear shift. Operating the engine at a lower speed after a shift may allow the engine to operate at a lower fuel consumption level as compared to if the transmission were shifted at higher vehicle speeds during a period when gaseous fuel is injected to the engine. The upshifting schedule for higher transmission gears may be adjusted similarly. Downshifting schedules may also be adjusted so that the transmission downshifts at a lower vehicle speed when operated with gaseous fuel direct injection as compared to when the engine and vehicle are operated at similar conditions with gaseous fuel port injection. For example, a third gear to second gear downshift may be scheduled at a vehicle speed of 42 Kph when the engine is operated with gaseous direct fuel injection and a vehicle speed of 48 Kph when the engine is operated with gaseous port fuel injection. The downshifting schedule for other transmission gears may be adjusted similarly.

The transmission torque converter lockup schedule is also adjusted based on the method of gaseous fuel injection. For example, if the engine switches from port gaseous fuel injection to gaseous direct injection, the vehicle speed (or engine speed) at which the torque converter lockup occurs is decreased. Thus, a torque converter lockup may be scheduled for a vehicle speed of 80 Kph when the engine receives directly injected gaseous fuel while a transmission is in fourth gear. On the other hand, the torque converter lockup may be scheduled for a vehicle speed of 88 Kph when the engine receives port injected gaseous fuel while the transmission is in fourth gear. Decreasing the torque converter lockup speed allows the engine speed to be at a lower speed after the torque converter is locked which may improve fuel economy.

At **714**, method **700** judges whether or not a level or amount of fuel in the liquid fuel storage tank is less than (L.T.) a threshold level or amount. If so, the answer is yes and method **700** proceeds to **716**. Otherwise, the answer is no and method **700** proceeds to **718**.

At **716**, method **700** deactivates injection of liquid fuel. Liquid fuel may be deactivated until the liquid fuel tank is refilled or until pressure of gaseous fuel stored in the gaseous fuel tank is less than a second threshold level. Method **700** proceeds to **710** after liquid fuel is deactivated.

At **718**, method **700** judges whether or not a desired driver demand torque is greater than (G.T.) a threshold torque and gaseous port injected fuel is activated. If method **700** judges that port or intake manifold gaseous fuel injection is active and driver demand torque is greater than the threshold

torque, the answer is yes and method **700** proceeds to **720**. Otherwise, the answer is no and method **700** proceeds to **722**.

At **720**, method **700** increases a liquid fraction of a total fuel amount delivered to the engine. For example, if a total fuel amount delivered to the engine is comprised of 10% by mass of liquid fuel and 90% of mass gaseous fuel, the mass of liquid fuel may be increased to 15% of the total fuel mass. In other examples, the fuel fraction may be increased based on energy density rather than by fuel mass. For example, if liquid fuel supplies 20% of energy supplied to the engine in an engine cycle, the liquid energy supplied may be increased to 25% of energy supplied to the engine during an engine cycle (e.g., two engine revolutions for a four stroke engine). In one example, the liquid fraction of fuel supplied to the engine during a cylinder cycle is increased by a predetermined amount (e.g., 5%). Method **700** proceeds to exit after the liquid fuel fraction has been increased.

At **722**, method **700** judges whether the engine is presently knock limited. In one example, method **700** judges that the engine is knock limited based on predetermined engine knock limits that are stored based on engine speed and load. In other examples, method **700** judges whether or not the engine is presently knock limited based on output of a knock sensor. If method **700** judges that the engine is knock limited the answer is yes and method **700** proceeds to **723**. Otherwise, the answer is no and method **700** proceeds to **724**.

At **723**, method **700** increases the fraction of fuel that has a higher effective octane supplied to the engine. The effective octane of the fuel is based on the fuel injected and the injection path (e.g., direct injection or port injection). For example, a gaseous fuel may have an effective port injection octane of 130 RON and an effective direct injection octane of 140 RON due to improved charge cooling via direct injection and injection timing. On the other hand, the liquid fuel may have an octane rating of 89. Consequently, the fraction of gaseous fuel supplied to the engine is increased. For example, the gaseous fuel fraction may be increased from 80% to 85% of the total amount of fuel supplied to the engine during an engine cycle. Method **700** proceeds to exit after the fraction of fuel having a higher effective octane is increased.

The base fractions of each fuel supplied to the engine may be empirically determined and stored in memory. The base fuel fractions may be indexed based on engine speed and load or requested torque. For example, at 1500 RPM and 0.2 load, fuel supplied to the engine during an engine cycle may be comprised of 95% gaseous fuel and 5% liquid fuel. The gaseous fuel fraction may decrease as engine load increases and vice versa for the liquid fuel fraction.

At **724**, method **700** judges whether or not the relative cost of the respective fuels is known. In one example, the driver may input cost of liquid and gaseous fuels to controller **12** via a key pad or user interface. In other examples, the internet or a fuel pump may supply the fuel cost information to controller **12** via wireless interface. If the cost of each fuel supplied to the engine is known, the answer is yes and method **700** proceeds to **725**. Otherwise, the answer is no and method **700** proceeds to **726**.

At **725**, method **700** increases the fraction of fuel of lower cost fuel supplied to the engine. In one example, the cost of each fuel is converted to a gasoline gallon equivalent (GGE) based on the cost and energy in the fuel supplied to the engine. For example, it may be determined that a predetermined mass of gaseous fuel has an equivalent amount of energy as a gallon of gasoline. The predetermined gaseous fuel mass is then multiplied by the cost per unit mass to

determine the GGE price of the gaseous fuel. Once the GGE cost of each fuel is determined, method **700** increases the fractional amount of the low cost fuel supplied to the engine relative to the total fuel supplied to the engine. The fractional increase of low cost fuel supplied to the engine may be a predetermined amount up to a predetermined limit. For example, if gaseous fuel supplied to the engine is 10% less expensive than liquid fuel supplied to the engine, the gaseous fuel fraction may be increased by 5% up to a maximum increase of 20%. Method **700** proceeds to exit after the fraction of fuel having a lower cost is increased.

At **726**, method **700** judges whether or not the engine is presently throttled by more than a predetermined amount. In one example, method **700** judges whether or not the engine is throttled based on intake manifold pressure. If intake manifold pressure is less than a predetermined amount, it may be determined that the engine is throttled by more than a predetermined amount. If method **700** judges that the engine is throttle by more than a predetermined amount, the answer is yes and method **700** proceeds to **727**. Otherwise, the answer is no and method **700** proceeds to **728**.

At **727**, method **700** increases the fraction of gaseous port or intake manifold injected fuel supplied to the engine. Further, if the engine is presently being directly injected with gaseous fuel, port gaseous fuel injection may be activated. The fraction of port or intake manifold injected gaseous fuel may be increased by a predetermined amount (e.g., 5%) up to a threshold amount (e.g., 95%). By increasing the portion or fraction of port or intake manifold injected fuel, the engine may be operated less throttled so that engine pumping losses may be reduced. Note that the engine throttle may be opened as the fraction of port injected gaseous fuel is increased to provide equivalent torque and air-fuel ratio as compared to when the engine is direct gaseous fuel injected at the same engine speed and torque demand. Method **700** proceeds to exit after the gaseous port injection fuel fraction is increased.

At **728**, method **700** judges whether or not the carbon intensities of fuel stored on board the vehicle are known. The carbon intensity may be referred to as a mass of carbon dioxide emissions from a fuel relative to the energy in the fuel. For example, CNG is typically 62 g CO₂ per megajoule of energy while gasoline is typically 73 g CO₂ per megajoule of energy. In one example, the carbon intensity of fuels stored in the vehicle may be based on information supplied from the internet or a fuel filling station via wireless transmitter. If method **700** judges that the relative carbon intensities of the fuels stored within the vehicle are known, the answer is yes and method **700** proceeds to **729**. Otherwise, method **700** exits.

At **729**, method **700** increases the fraction of the lower carbon fuel supplied to the engine. For example, if the liquid fuel stored in the vehicle has X grams of carbon per gram of fuel and a gaseous fuel stored in the vehicle has X-Y grams of carbon, method **700** increases the fraction gaseous fuel fraction in a total amount of fuel supplied during an engine cycle. The fraction of gaseous fuel may be increased by a predetermined amount up to a limit. For example, the fraction of gaseous fuel supplied to the engine may increase from 25% to 35% during an engine cycle. Method **700** proceeds to exit after the fraction of low carbon fuel supplied to the engine during an engine cycle is increased.

Thus, the method of FIG. 7 provides for an engine operating method, comprising: directly injecting a gaseous fuel to an engine in response to pressure of gaseous fuel stored in a fuel tank exceeding a threshold level; and increasing an idle speed of the engine and injecting the

gaseous fuel to an engine air intake in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level.

In one example, the method includes where the engine is operated at a first idle speed while fuel is directly injected to the engine, where the engine is operated at a second idle speed while fuel is engine air intake injected, and where the first idle speed is less than the second idle speed. The method further comprises decreasing the idle speed of the engine in response to the pressure of gaseous fuel stored in the fuel tank being greater than the threshold level. The method further comprises increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to a desired engine torque exceeding a threshold level. The method further comprises increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to activating an engine air intake injector. The method includes where the engine air intake injecting the gaseous fuel includes injecting gaseous fuel into an engine intake manifold. The method includes where the engine air intake injecting the gaseous fuel includes injecting gaseous fuel into a cylinder intake port.

The method of FIG. 7 also provides for an engine operating method, comprising: directly injecting a gaseous fuel to an engine in response to pressure of gaseous fuel stored in a fuel tank exceeding a threshold level; and increasing an engine idle speed, adjusting torque converter clutch locking, and engine air intake injecting the gaseous fuel to the engine in response to the amount of gaseous fuel stored in the fuel tank being less than the threshold level. The method includes where the torque converter clutch is held in an open state without locking while engine air intake injecting the gaseous fuel. The method includes where directly injecting the gaseous fuel is deactivated while engine air intake injecting the gaseous fuel. The method also includes where engine air intake injecting the gaseous fuel is deactivated while directly injecting the gaseous fuel. The method includes where adjusting the torque converter clutch locking includes increasing a vehicle speed where the torque converter clutch is locked in response to the amount of gaseous fuel stored in the fuel tank being less than the threshold level. The method further comprises increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to a desired engine torque exceeding a threshold level.

Referring now to FIG. 8, a method for operating a vehicle is shown. The method may be incorporated into the system of FIGS. 1-2 and 4 as executable instructions stored in non-transitory memory. The method of FIG. 8 may provide the operating sequence shown in FIG. 6.

At **802**, method **800** determines engine and fuel tank conditions. Engine conditions may include but are not limited to engine speed, driver demand torque, present type of fuel injected, and engine temperature. Fuel tank conditions may include but are not limited to fuel level or amount of fuel stored in a fuel tank, fuel temperature, fuel pressure, fuel pump state (e.g., on or off), and fuel delivery path (e.g., port injection/direct injection). Method **800** proceeds to **804** after engine and fuel tank conditions are determined.

At **804**, method **800** judges whether or not gaseous fuel tank pressure is less than (L.T.) a first threshold pressure. The first threshold pressure may be indicative of fuel pressure to introduce fuel directly to a cylinder during a compression stroke of the cylinder. Further, the first threshold may vary with engine operating conditions (e.g., engine speed and demand torque). If method **800** judges that the

gaseous fuel tank pressure is less than the first threshold pressure, the answer is yes and method **800** proceeds to **806**. Otherwise, the answer is no and method **800** proceeds to **808**.

At **806**, method **800** activates a gaseous fuel pump to increase pressure of fuel supplied to direct injectors. The gaseous fuel pump is in pneumatic communication with the gaseous fuel storage tank as shown in FIG. 4. The gaseous fuel pump may supply gaseous fuel to direct injectors via a control valve. In one example, the control valve is a three-way valve that selects between gaseous fuel storage tank output and gaseous fuel pump output. The gaseous fuel pump may supply pressurized gaseous fuel to an accumulator tank and output from the accumulator tank may be directed to direct fuel injectors. The gaseous fuel pump may be selectively activated and deactivated depending on pressure at the fuel pump outlet and/or pressure in the accumulator. In examples where the gaseous fuel pump has variable pumping capacity, the pump's capacity may be reduced when pressure in the accumulator is at or above a desired pressure. The gaseous fuel pump's capacity may be increased when pressure in the accumulator is less than desired. Pressure at the outlet of the accumulator may be regulated via a pressure regulator.

In some examples, the gaseous fuel pump may be sized to operate the engine at rated load. However, in other examples, the gaseous fuel pump may be sized with an output capacity that is less than the fuel flow rate to operate the engine at full load with a slightly rich air-fuel ratio (e.g., 0.15 air-fuel ratio richer than stoichiometry).

The transmission gear shifting schedule may also be adjusted based on the activation status of the gaseous fuel pump and amount of gaseous fuel stored in the gaseous fuel storage tank. For example, if the gaseous fuel pump is activated, the engine speed at which each transmission gear upshift occurs may be decreased to avoid exceeding the flow capacity of the gaseous fuel pump. Thus, a first gear to second gear upshift at a first engine torque demand may be decreased from a vehicle speed of 22 Kilometers per hour (Kph) to a vehicle speed of 16 Kph so that the gaseous fuel flow rate may be reduced. In some examples, the throttle opening amount or cam timing or boost pressure may be adjusted to limit engine air flow and torque so that a smaller capacity gaseous fuel pump may supply gaseous fuel to the engine. The upshifting schedule for higher transmission gears may be adjusted similarly. Downshifting schedules may also be adjusted so that the transmission downshifts at a lower vehicle speed when operated with the gaseous fuel pump as compared to when the engine may be supplied gaseous fuel without use of a gaseous fuel pump. For example, a third gear to second gear downshift may be scheduled at a vehicle speed of 42 Kph when the engine is operated with the gaseous fuel pump and at a vehicle speed of 48 Kph when the engine is operated without the gaseous fuel pump. The downshifting schedule for other transmission gears may be adjusted similarly.

The transmission torque converter lockup schedule may also be adjusted based on the operating state of the gaseous fuel pump. For example, if the engine switches from operating without the gaseous fuel pump to operating with the gaseous fuel pump, the vehicle speed (or engine speed) at which the torque converter lockup occurs may be decreased to avoid exceeding the flow capacity of the gaseous fuel pump. Thus, a torque converter lockup may be scheduled for a vehicle speed of 80 Kph when the engine receives gaseous fuel via the gaseous fuel pump while a transmission is in fourth gear. On the other hand, the torque converter lockup

may be scheduled for a vehicle speed of 88 Kph when the engine receives gaseous fuel directly from the gaseous fuel tank while the transmission is in fourth gear. Decreasing the torque converter lockup speed allows the engine speed to be at a lower speed after the torque converter is locked so that the engine may be operating at a lower speed to reduce gaseous fuel flow rate.

At **810**, method **800** judges whether or not gaseous fuel tank pressure is less than (L.T.) a second threshold pressure. The second threshold pressure is less than the first threshold pressure. If so, the answer is yes and method **800** proceeds to **812**. Otherwise, the answer is no and method **800** proceeds to **814**.

At **812**, method **800** uses fuel remaining in the gaseous and liquid fuel tanks to extend the vehicle's driving range even when the amount of fuel stored in the gaseous and liquid fuel tanks is low. In one example, method **800** selects a fuel type to operate the engine with based on the fuel tank that has the greatest amount of stored fuel. For example, if the gaseous fuel tank is one quarter full and the liquid fuel tank is one eighth full, method **800** selects gaseous fuel to operate the engine on and injects the gaseous fuel via port gaseous fuel injectors. In one example, method **800** operates the engine with the selected fuel for a predetermined duration (e.g., amount of time or travel distance) and then switches over to the previously unselected fuel. For example, if the gaseous fuel is selected first, the engine operates with the gaseous fuel for ten minutes and then switches over to operating the engine using liquid fuel for ten minutes and then the engine is operated for another ten minutes with gaseous fuel. If the engine loses power while operating with one fuel, the engine switches over to the other fuel and continues to operate on the selected fuel until the vehicle is refueled or is out of fuel. Method **800** proceeds to exit after selecting fuel to operate the engine with.

At **808**, method **800** deactivates the gaseous fuel pump and supplies gaseous fuel to the engine without a gaseous fuel pump and directly from the gaseous fuel storage tank. Method **800** proceeds to **814** after direct gaseous fuel injectors are activated.

Further, when method **800** switches off the gaseous fuel pump, method **800** may adjust transmission gear shift schedules and torque converter lockup scheduling. In particular, the transmission gear shifting schedule may also be adjusted based on the operating state of the gaseous fuel pump. Specifically, if the engine switches from operating with the gaseous fuel pump to operating without the gaseous fuel pump, the vehicle speed at which each transmission gear upshift occurs may be increased. Thus, a first gear to second gear upshift at a first engine torque demand may be increased from a vehicle speed of 16 Kilometers per hour (Kph) to a vehicle speed of 22 Kph so that engine power output may be increased if desired. The upshifting schedule for higher transmission gears may be adjusted similarly. The throttle opening amount may be allowed to open to its full extent since the gaseous fuel pump is not limiting fuel flow to the engine. Downshifting schedules may also be adjusted so that the transmission downshifts at a higher vehicle speed when operated without the gaseous fuel pump as compared to when the engine and vehicle are operated at similar conditions with the gaseous fuel pump. For example, a third gear to second gear downshift may be scheduled at a vehicle speed of 48 Kph when the engine is operated without the gaseous fuel pump and at a vehicle speed of 42 Kph when the engine is operated with the gaseous fuel pump. The downshifting schedule for other transmission gears may be adjusted similarly.

The transmission torque converter lockup schedule may also be adjusted based on stopping the gaseous fuel pump. For example, if the engine switches from operating the gaseous fuel pump to operating the engine without the gaseous fuel pump, the vehicle speed at which the torque converter lockup occurs may be increased. Thus, a torque converter lockup may be scheduled for a vehicle speed of 88 Kph when the engine receives gaseous fuel without the gaseous fuel pump while a transmission is in fourth gear. On the other hand, the torque converter lockup may be scheduled for a vehicle speed of 80 Kph when the engine receives gaseous fuel from the gaseous fuel pump while the transmission is in fourth gear. Increasing the torque converter lockup speed allows the engine to be at a higher speed after the torque converter is locked so that the engine may be operating at an engine speed where the engine's torque capacity is increased.

At **814**, method **800** judges whether or not a level or amount of fuel in the liquid fuel storage tank is less than (L.T.) a threshold level or amount. If so, the answer is yes and method **800** proceeds to **816**. Otherwise, the answer is no and method **800** proceeds to **818**.

At **816**, method **800** deactivates injection of liquid fuel. Liquid fuel may be deactivated until the liquid fuel tank is refilled or until pressure of gaseous fuel stored in the gaseous fuel tank is less than a second threshold level. Method **800** proceeds to **810** after liquid fuel is deactivated.

At **818**, method **800** judges whether or not a gaseous fuel flow rate is greater than (G.T.) a threshold flow and gaseous fuel pump is activated. If method **800** judges that the gaseous fuel flow rate is greater than the threshold rate and that the gaseous fuel pump is active, the answer is yes and method **800** proceeds to **820**. Otherwise, the answer is no and method **800** proceeds to **822**.

At **820**, method **800** increases a liquid fraction of a total fuel amount delivered to the engine. In one example, the liquid fraction of fuel supplied to the engine during a cylinder cycle is increased by a predetermined amount (e.g., 5%). Method **800** proceeds to exit after the liquid fuel fraction has been increased.

At **822**, method **800** judges whether the engine is presently knock limited. In one example, method **800** judges that the engine is knock limited based on predetermined engine knock limits that are stored based on engine speed and load. In other examples, method **800** judges whether or not the engine is presently knock limited based on output of a knock sensor. If method **800** judges that the engine is knock limited the answer is yes and method **800** proceeds to **823**. Otherwise, the answer is no and method **800** proceeds to **824**.

At **823**, method **800** increases the fraction of fuel that has a higher effective octane supplied to the engine. The effective octane of the fuel is based on the fuel injected and the injection path (e.g., direct injection or port injection). For example, a gaseous fuel may have an effective port injection octane of 130 RON and an effective direct injection octane of 140 RON due to improved charge cooling via direct injection and injection timing. On the other hand, the liquid fuel may have an octane rating of 89. Consequently, the fraction of gaseous fuel supplied to the engine is increased. For example, the gaseous fuel fraction may be increased from 80% to 85% of the total amount of fuel supplied to the engine during an engine cycle. Method **800** proceeds to exit after the fraction of fuel having a higher effective octane is increased.

At **824**, method **800** judges whether or not the relative cost of the respective fuels is known. In one example, the driver may input cost of liquid and gaseous fuels to con-

troller **12** via a key pad or user interface. In other examples, the internet or a fuel pump may supply the fuel cost information to controller **12** via wireless interface. If the cost of each fuel supplied to the engine is known, the answer is yes and method **800** proceeds to **825**. Otherwise, the answer is no and method **800** proceeds to **826**.

At **825**, method **800** increases the fraction of fuel of lower cost fuel supplied to the engine. In one example, the cost of each fuel is converted to a gasoline gallon equivalent (GGE) based on the cost and energy in the fuel supplied to the engine. For example, it may be determined that a predetermined mass of gaseous fuel has an equivalent amount of energy as a gallon of gasoline. The predetermined gaseous fuel mass is then multiplied by the cost per unit mass to determine the GGE price of the gaseous fuel. Once the GGE cost of each fuel is determined, method **800** increases the fractional amount of the low cost fuel supplied to the engine relative to the total fuel supplied to the engine. The fractional increase of low cost fuel supplied to the engine may be a predetermined amount up to a predetermined limit. For example, if gaseous fuel supplied to the engine is 10% less expensive than liquid fuel supplied to the engine, the gaseous fuel fraction may be increased by 5% up to a maximum increase of 20%. Method **800** proceeds to exit after the fraction of fuel having a lower cost is increased.

At **826**, method **800** judges whether or not the engine is presently throttled by more than a predetermined amount. In one example, method **800** judges whether or not the engine is throttled based on intake manifold pressure. If intake manifold pressure is less than a predetermined amount, it may be determined that the engine is throttled by more than a predetermined amount. If method **800** judges that the engine is throttle by more than a predetermined amount, the answer is yes and method **800** proceeds to **827**. Otherwise, the answer is no and method **800** proceeds to **828**.

At **827**, method **800** increases the fraction of gaseous port or intake manifold injected fuel supplied to the engine. Further, if the engine is presently being directly injected with gaseous fuel, port gaseous fuel injection may be activated. The fraction of port or intake manifold injected gaseous fuel may be increased by a predetermined amount (e.g., 5%) up to a threshold amount (e.g., 95%). By increasing the portion or fraction of port or intake manifold injected fuel, the engine may be operated less throttled so that engine pumping losses may be reduced. Note that the engine throttle may be opened as the fraction of port injected gaseous fuel is increased to provide equivalent torque and air-fuel ratio as compared to when the engine is direct gaseous fuel injected at the same engine speed and torque demand. Method **800** proceeds to exit after the gaseous port injection fuel fraction is increased.

At **828**, method **800** judges whether or not the carbon intensities of fuel stored on board the vehicle are known. The carbon intensity may be referred to as a mass of carbon dioxide emissions from a fuel relative to the energy in the fuel. In one example, the carbon intensity of fuels stored in the vehicle may be based on information supplied from the internet or a fuel filling station via wireless transmitter. If method **800** judges that the relative carbon intensities of the fuels stored within the vehicle are known, the answer is yes and method **800** proceeds to **829**. Otherwise, method **800** exits.

At **829**, method **800** increases the fraction of the lower carbon fuel supplied to the engine. For example, if the liquid fuel stored in the vehicle has X grams of carbon per gram of fuel and a gaseous fuel stored in the vehicle has X-Y grams of carbon, method **800** increases the fraction gaseous fuel

fraction in a total amount of fuel supplied during an engine cycle. The fraction of gaseous fuel may be increased by a predetermined amount up to a limit. Method **800** proceeds to exit after the fraction of low carbon fuel supplied to the engine during an engine cycle is increased.

Thus, the method of FIG. **8** provides for an engine operating method, comprising: directly injecting a gaseous fuel to an engine in response to a pressure of a gaseous fuel stored in a fuel tank exceeding a threshold pressure; and activating a gaseous fuel pump only when the pressure of gaseous fuel stored in the fuel tank is less than or equal to the threshold level. The method includes where a direct injector is supplied fuel from the fuel tank without a gaseous fuel pump in response to the pressure of gaseous fuel stored in the fuel tank exceeding the threshold pressure.

In one example, the method includes where the direct injector is supplied fuel from the gaseous fuel pump in response to the pressure of the gaseous fuel stored in the fuel tank being less than or equal to the threshold level. The method further comprises a three-way valve and adjusting a state of the three-way valve in response to the pressure of gaseous fuel stored in the fuel tank. The method further comprises regulating output of the gaseous fuel pump to a desired pressure. The method includes where the threshold pressure is varied with engine operating conditions.

The method of FIG. **8** also provides for an engine operating method, comprising: directly injecting a gaseous fuel to an engine in response to a pressure of a gaseous fuel stored in a fuel tank exceeding a threshold pressure; and activating a gaseous fuel pump, adjusting torque converter clutch locking, and adjusting a transmission gear shift schedule in response to the pressure of the gaseous fuel stored in the fuel tank being less than or equal to the threshold pressure. The method includes where adjusting the transmission gear shift schedule includes adjusting a vehicle speed at which a transmission shifts gears. The method includes where the vehicle speed at which the transmission shifts gears is decreased in response to activating the gaseous fuel pump. The method includes where adjusting torque converter clutch locking includes adjusting a vehicle speed at which the torque converter clutch locks.

In some examples, the method includes where the vehicle speed at which the torque converter locks is decreased in response to activating the gaseous fuel pump. The method further comprises adjusting an operating state of a three-way valve that is in fluidic communication with the gaseous fuel pump and the fuel tank. The method includes where the gaseous fuel pump supplies fuel to a direct injector. The method includes where the direct injector is supplied gaseous fuel via the fuel tank without the fuel passing through the gaseous fuel pump.

As will be appreciated by one of ordinary skill in the art, method described in FIGS. **7** and **8** may represent one or more of any number of processing strategies such as event-driven, interrupt-driven, multi-tasking, multi-threading, and the like. As such, various steps or functions illustrated may be performed in the sequence illustrated, in parallel, or in some cases omitted. Likewise, the order of processing is not necessarily required to achieve the objects, features, and advantages described herein, but is provided for ease of illustration and description. Although not explicitly illustrated, one of ordinary skill in the art will recognize that one or more of the illustrated steps or functions may be repeatedly performed depending on the particular strategy being used. Further, the described actions, operations, methods, and/or functions may graphically represent code to be pro-

grammed into non-transitory memory of the computer readable storage medium in the engine control system.

This concludes the description. The reading of it by those skilled in the art would bring to mind many alterations and modifications without departing from the spirit and the scope of the description. For example, I3, I4, I5, V6, V8, V10, and V12 engines operating in natural gas, gasoline, diesel, or alternative fuel configurations could use the present description to advantage.

The invention claimed is:

1. An engine operating method, comprising:

directly injecting a gaseous fuel to an engine in response to a pressure of gaseous fuel stored in a fuel tank exceeding a threshold level;

increasing an idle speed of the engine and injecting the gaseous fuel to an engine air intake in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level; and

adjusting a transmission shifting schedule in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level.

2. The method of claim **1**, where the idle speed is a speed at which the engine rotates when driver demand torque is zero and when engine speed is adjusted to a substantially constant speed.

3. The method of claim **1**, where the engine is operated at a first idle speed while fuel is directly injected to the engine, where the engine is operated at a second idle speed while fuel is engine air intake injected, and where the first idle speed is less than the second idle speed.

4. The method of claim **3**, further comprising decreasing the idle speed of the engine in response to the pressure of gaseous fuel stored in the fuel tank being greater than the threshold level.

5. The method of claim **1**, further comprising directly injecting a liquid fuel to the engine via an injector positioned to inject fuel directly into a cylinder of the engine; and increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to a desired engine torque exceeding a threshold level.

6. The method of claim **1**, further comprising directly injecting a liquid fuel to the engine via an injector positioned to inject fuel directly into a cylinder of the engine; and increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to activating an engine air intake injector.

7. The method of claim **1**, where injecting the gaseous fuel to the engine air intake includes injecting gaseous fuel into an engine intake manifold.

8. The method of claim **1**, where injecting the gaseous fuel to the engine air intake includes injecting gaseous fuel into an engine intake port.

9. An engine operating method, comprising:

directly injecting a gaseous fuel to an engine in response to a pressure of gaseous fuel stored in a fuel tank exceeding a threshold level;

increasing an engine idle speed, adjusting torque converter clutch locking, and engine air intake injecting the gaseous fuel to the engine in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level; and

adjusting a transmission shifting schedule in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level.

10. The method of claim **9**, where a torque converter clutch is held in an open state without locking while engine air intake injecting the gaseous fuel.

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11. The method of claim 9, where directly injecting the gaseous fuel is deactivated while engine air intake injecting the gaseous fuel.

12. The method of claim 11, where adjusting the torque converter clutch locking includes increasing a vehicle speed where a torque converter clutch is locked in response to the pressure of gaseous fuel stored in the fuel tank being less than the threshold level.

13. The method of claim 12, further comprising increasing a fraction of liquid fuel in a total amount of fuel supplied to the engine during an engine cycle in response to a desired engine torque exceeding a threshold level.

14. The method of claim 9, where engine air intake injecting the gaseous fuel is deactivated while directly injecting the gaseous fuel.

15. A vehicle system, comprising:
 an engine with direct fuel injection;
 a transmission;
 a torque converter coupled between the engine and transmission;
 a gaseous fuel tank; and
 a controller including non-transitory instructions for increasing engine idle speed and adjusting a transmission shifting schedule in response to a pressure of gaseous fuel stored in the gaseous fuel tank being less than a threshold level, the controller further adjusting

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an engine intake throttle opening amount or cam timing or boost pressure to limit engine air flow and torque.

16. The vehicle system of claim 15, further comprising non-transitory instructions for adjusting a torque converter lock-up schedule in response to the pressure of gaseous fuel stored in the gaseous fuel tank being less than the threshold level.

17. The vehicle system of claim 15, further comprising a step ratio transmission, and where adjusting the transmission shifting schedule increases a vehicle speed at which the step ratio transmission is shifted.

18. The vehicle system of claim 17, further comprising non-transitory instructions for adjusting a torque at which the step ratio transmission is shifted.

19. The vehicle system of claim 18, where the torque at which the step ratio transmission is shifted is reduced in response to the pressure of gaseous fuel stored in the gaseous fuel tank being less than the threshold level.

20. The vehicle system of claim 15, further comprising non-transitory instructions for ceasing direct injection of a gaseous fuel to the engine and starting engine intake injection of the gaseous fuel in response to the pressure of gaseous fuel stored in the gaseous fuel tank being less than the threshold level.

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