

US007896590B2

(12) **United States Patent**
Miller et al.

(10) **Patent No.:** **US 7,896,590 B2**
(45) **Date of Patent:** **Mar. 1, 2011**

(54) **CUTTER FOR ENGAGEMENT WITH AN ARBOR**

RE31,755 E 12/1984 Wanner et al.
4,775,269 A 10/1988 Brix
4,818,157 A 4/1989 Kouvelis

(75) Inventors: **Jeffery R. Miller**, Gaines, MI (US);
Gary D. Taphouse, Owosso, MI (US)

(73) Assignee: **Hougen Manufacturing, Inc.**, Swartz Creek, MI (US)

(Continued)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 796 days.

FOREIGN PATENT DOCUMENTS

DE 4222809 A1 1/1993

(21) Appl. No.: **11/924,559**

(Continued)

(22) Filed: **Oct. 25, 2007**

OTHER PUBLICATIONS

(65) **Prior Publication Data**

US 2009/0110500 A1 Apr. 30, 2009

Computer translation of DE 10100024 from EPO website, printed on Oct. 24, 2010.*

(51) **Int. Cl.**
B23B 51/04 (2006.01)

(Continued)

(52) **U.S. Cl.** **408/226**; 408/204; 408/240;
279/81; 279/83; 279/137

Primary Examiner—Daniel W Howell

(58) **Field of Classification Search** 279/83,
279/93, 81, 75, 137, 19.3–19.6, 71; 408/204,
408/226, 238, 239 R, 240; *B23B 31/107*,
B23B 51/04, *51/05*

(74) *Attorney, Agent, or Firm*—Howard & Howard Attorneys PLLC

See application file for complete search history.

(57)

ABSTRACT

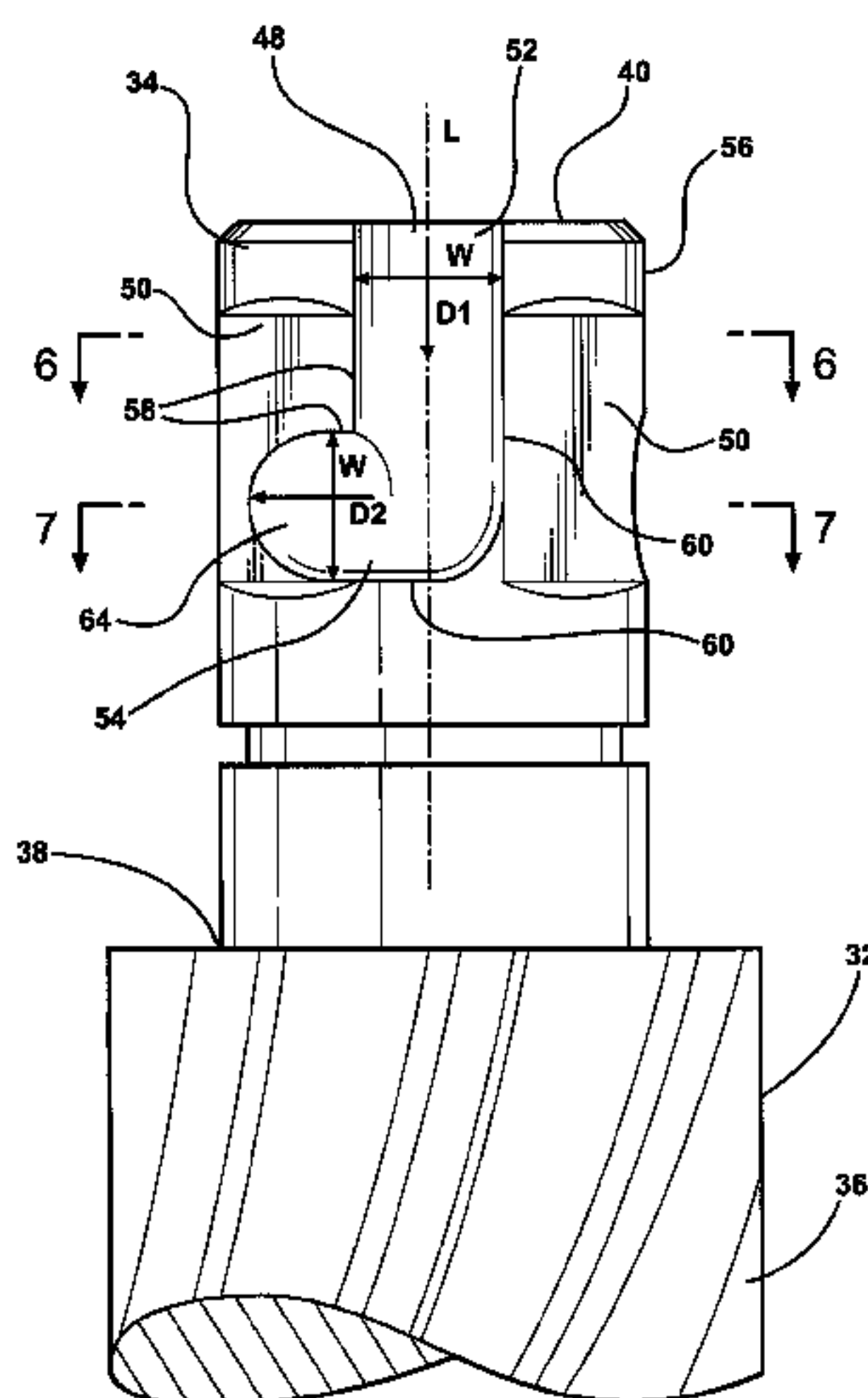
(56) **References Cited**

A cutter is for driven engagement with an arbor having one of an extending member and a pair of set screws. The cutter includes a shank portion having a central axis and a cutting portion extending from the shank portion along the central axis. A retaining recess is formed in the shank portion for selectively receiving the extending member of the arbor when the arbor is configured to have the driving device. A pair of flats are each formed in the shank portion for engaging the set screws when the arbor is configured to have the set screws. One of the flats extends in a first plane and the other of the flats extends in a second plane perpendicular to the first plane.

U.S. PATENT DOCUMENTS

2,039,855 A * 5/1936 Stone 279/79
2,079,460 A 5/1937 Marty
2,731,273 A 1/1956 Edens
2,823,040 A 2/1958 Fischer
2,906,541 A 9/1959 DuBois
2,960,343 A 11/1960 Elledge
2,974,965 A * 3/1961 Welles, Jr. 279/83
3,405,950 A 10/1968 Cox
3,747,946 A 7/1973 Edens
4,107,949 A 8/1978 Wanner et al.
4,188,041 A 2/1980 Soderberg

20 Claims, 8 Drawing Sheets



U.S. PATENT DOCUMENTS

5,167,475 A 12/1992 Miyanaga
 5,281,060 A 1/1994 Strange et al.
 5,427,482 A 6/1995 Asano et al.
 D360,641 S 7/1995 Asano
 5,429,457 A 7/1995 Asano et al.
 D361,579 S 8/1995 Asano
 D361,776 S 8/1995 Asano
 5,904,456 A * 5/1999 Chern 408/240
 6,280,123 B1 8/2001 Gill
 D560,699 S * 1/2008 Omi D15/139
 2005/0105981 A1 5/2005 Byrley et al.
 2006/0159531 A1 7/2006 Bortz et al.

FOREIGN PATENT DOCUMENTS

DE 10100024 A1 * 7/2002
 EP 0722798 A1 7/1996
 EP 1674182 A1 6/2006
 JP 06-315806 A 11/1994
 JP 09011007 A 1/1997
 JP 10-100009 A 4/1998
 JP 2000033509 A * 2/2000
 JP 2001-198716 A 7/2001
 JP 2002120218 A 4/2002
 WO WO 02064295 A2 8/2002
 WO WO 2004087357 A1 10/2004

WO WO 2007118626 A1 10/2007

OTHER PUBLICATIONS

English language abstract for DE4222809 extracted from espacenet.com database, dated Nov. 7, 2008.
 English language abstract for EP1674182 extracted from espacenet.com database, dated Nov. 7, 2008.
 English language translation and abstract for JP09-011007 extracted from PAJ database, dated Nov. 7, 2008, 31 pages.
 English language translation and abstract for JP2002-120218 extracted from PAJ database, dated Nov. 7, 2008, 26 pages.
 English language abstract for WO 2007/118626, 54 pages.
 PCT International Search Report for PCT/US2008/008045, dated Oct. 24, 2008, 5 pages.
 PCT International Search Report for PCT/US2008/008046, dated Oct. 24, 2008, 4 pages.
 Design U.S. Appl. No. 29/296,707, Miller et al., "Annular Cutter", filed Oct. 25, 2007, 4 pp.
 U.S. Appl. No. 11/924,551, Miller et al., "Cutter With Engagement for an Arbor", filed Oct. 25, 2007, 29 pp.
 English language translation and abstract for JP 06-315806 extracted from PAJ database, dated Dec. 17, 2010, 32 pages.
 English language translation and abstract for JP 10-100009 extracted from PAJ database, dated Dec. 17, 2010, 22 pages.
 English language translation and abstract for JP 2001-198716 extracted from PAJ database, dated Dec. 17, 2010, 37 pages.

* cited by examiner

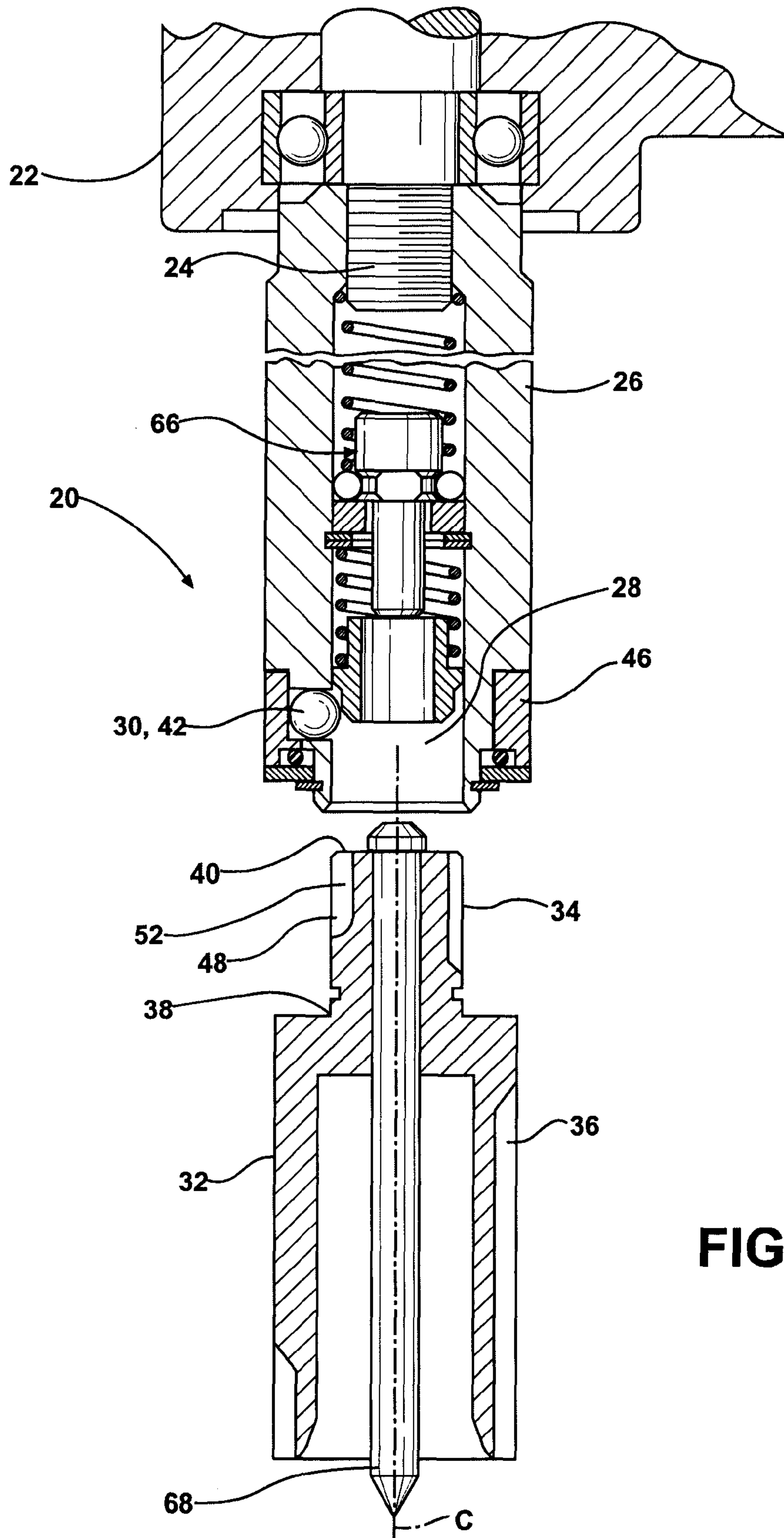


FIG - 1

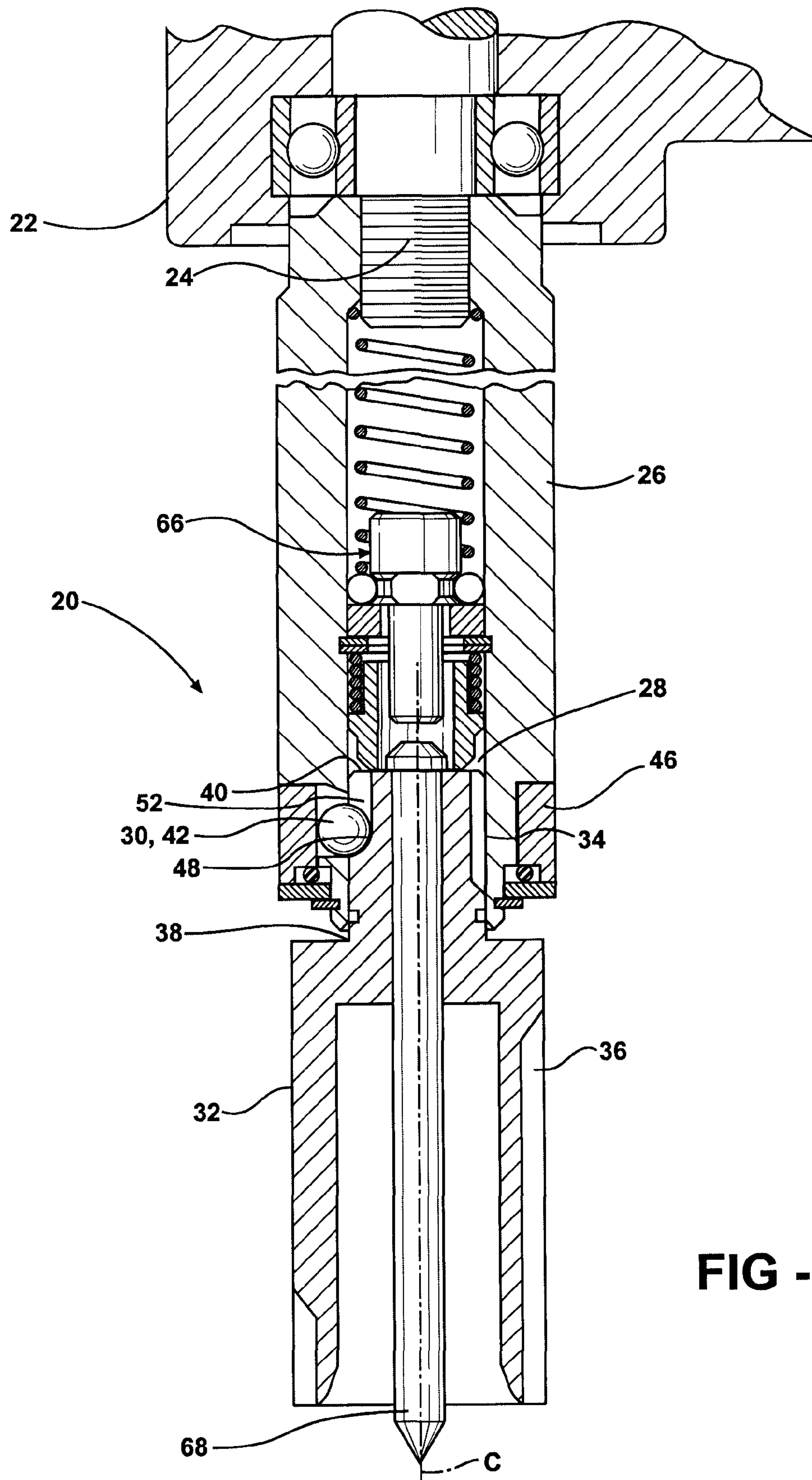


FIG - 2

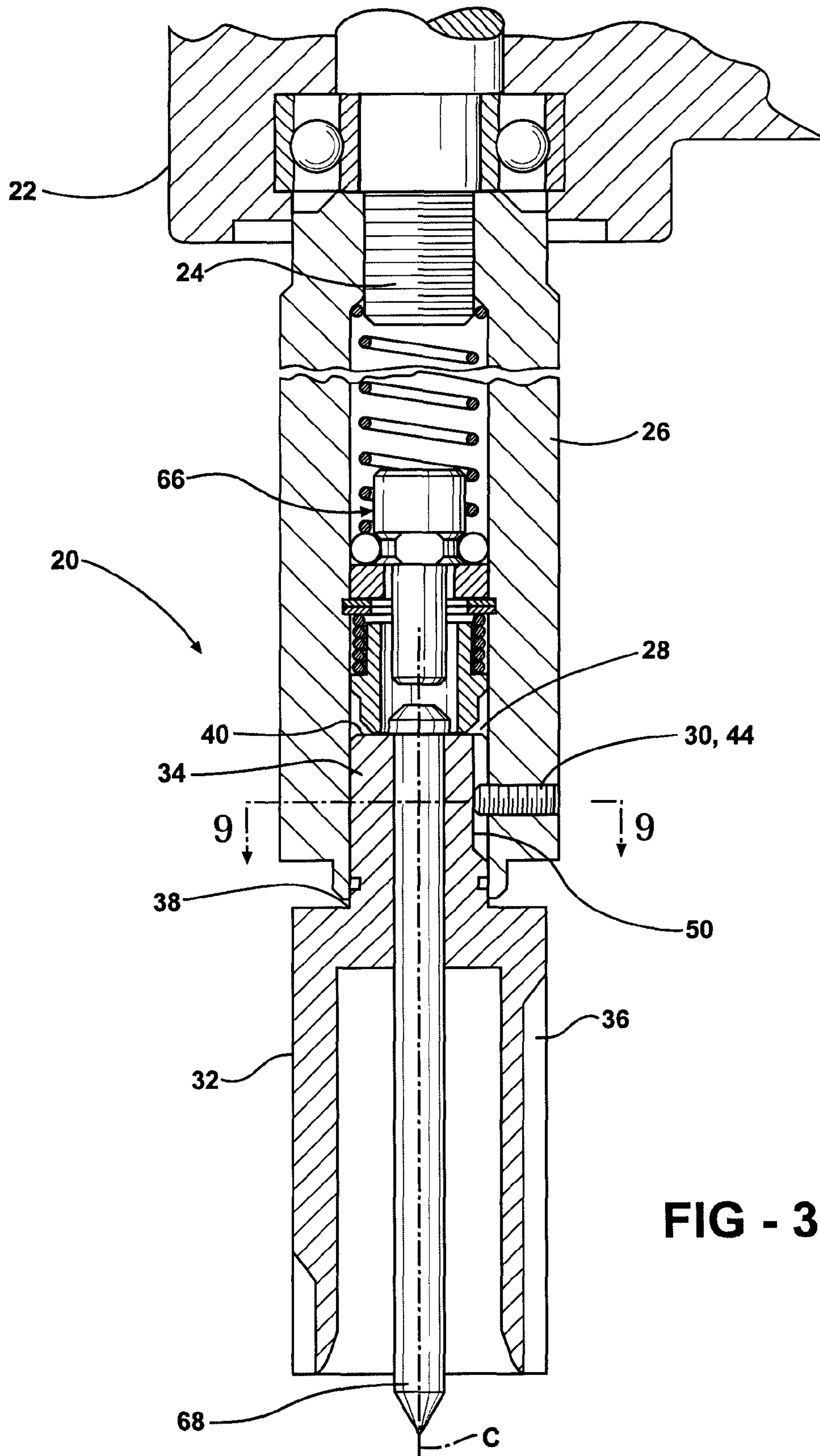


FIG - 3

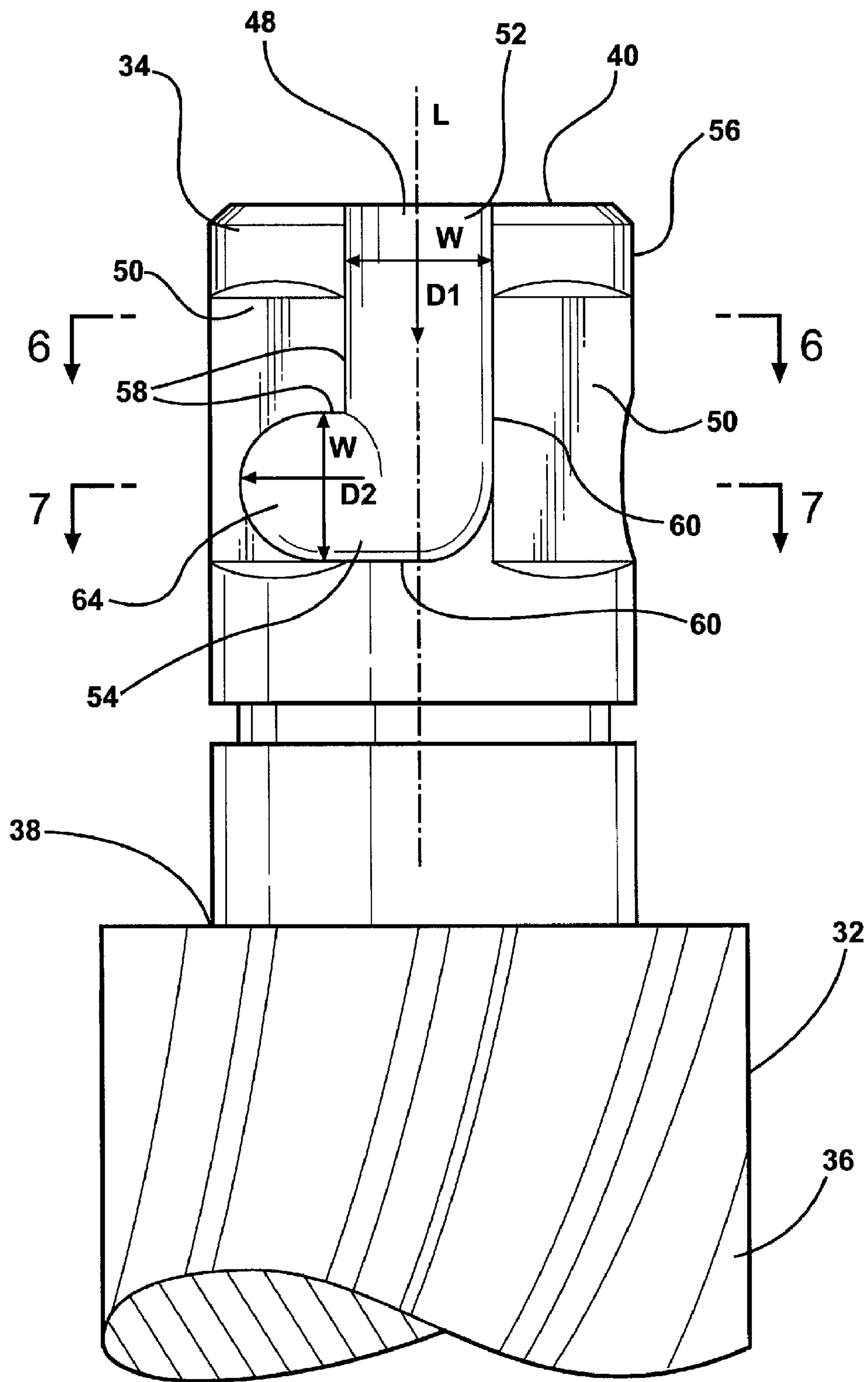


FIG - 4

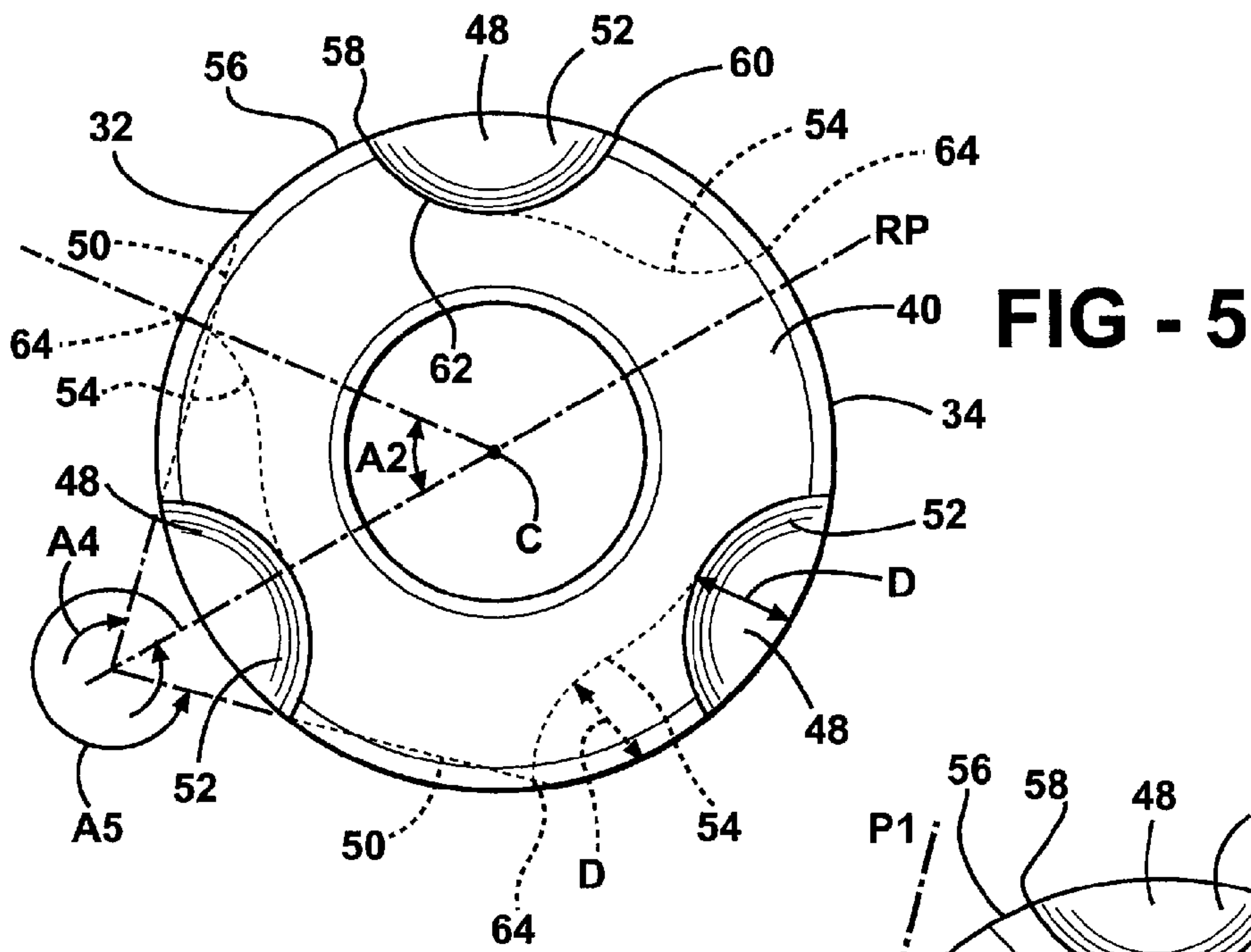


FIG - 6

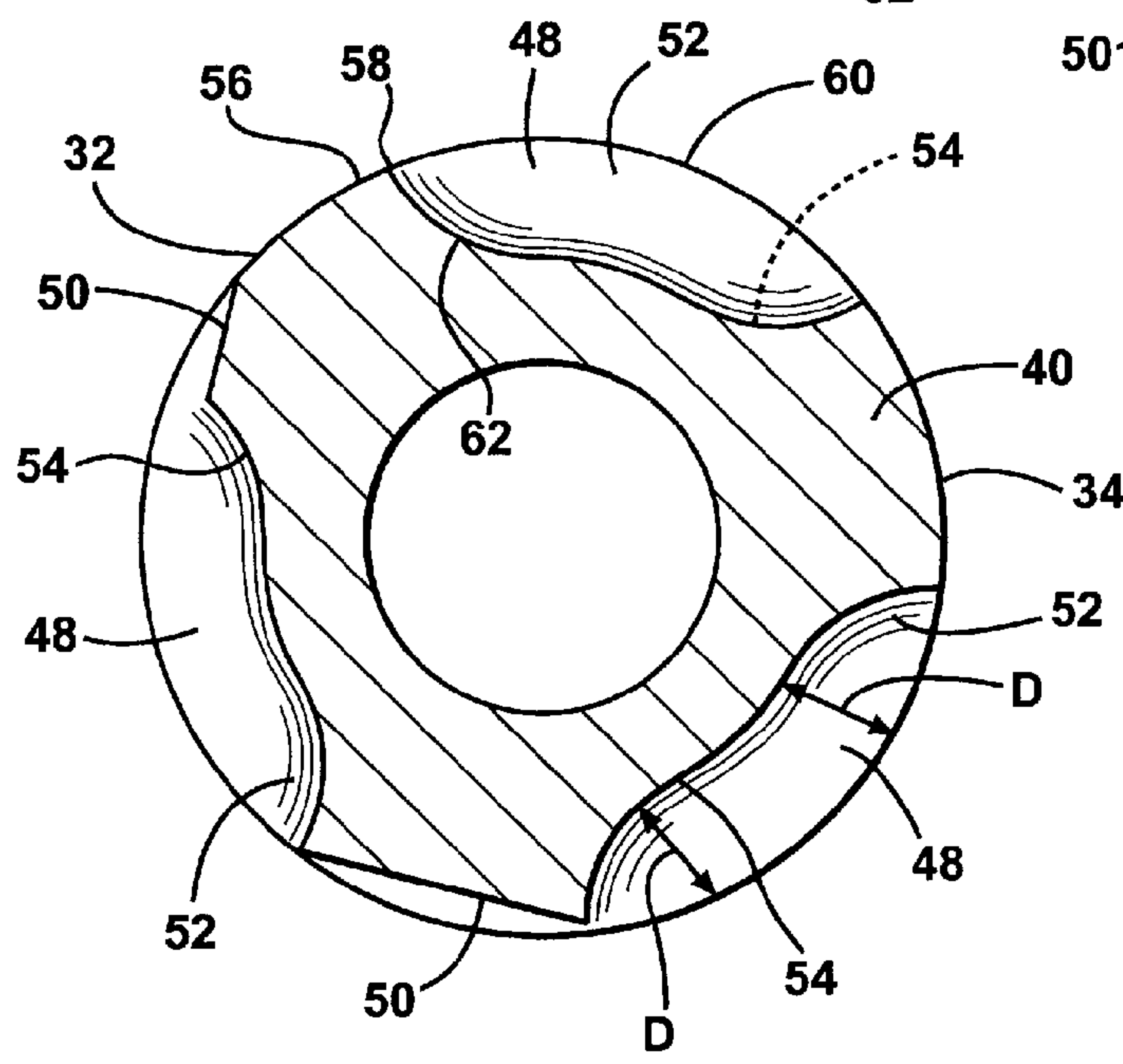
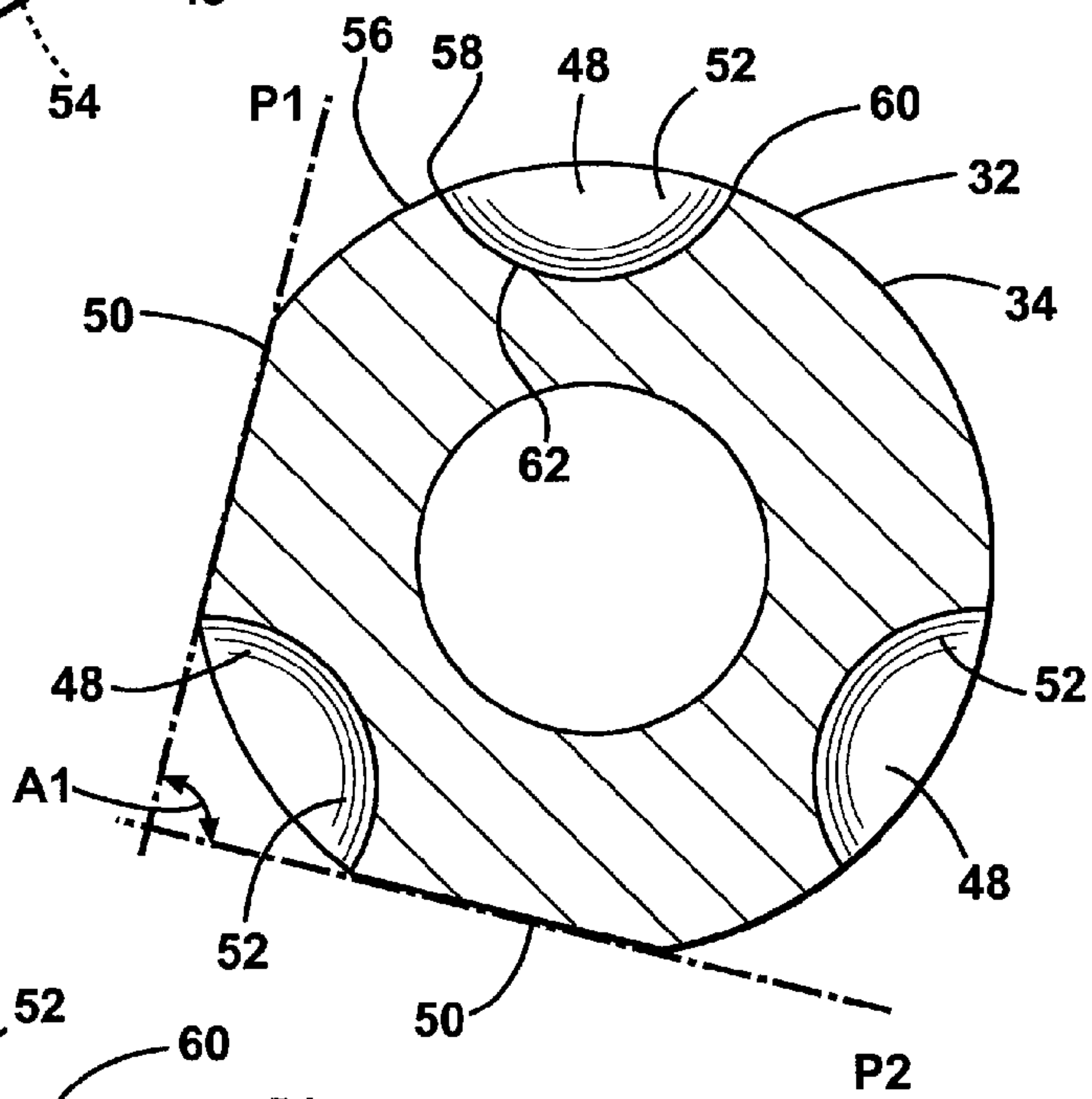
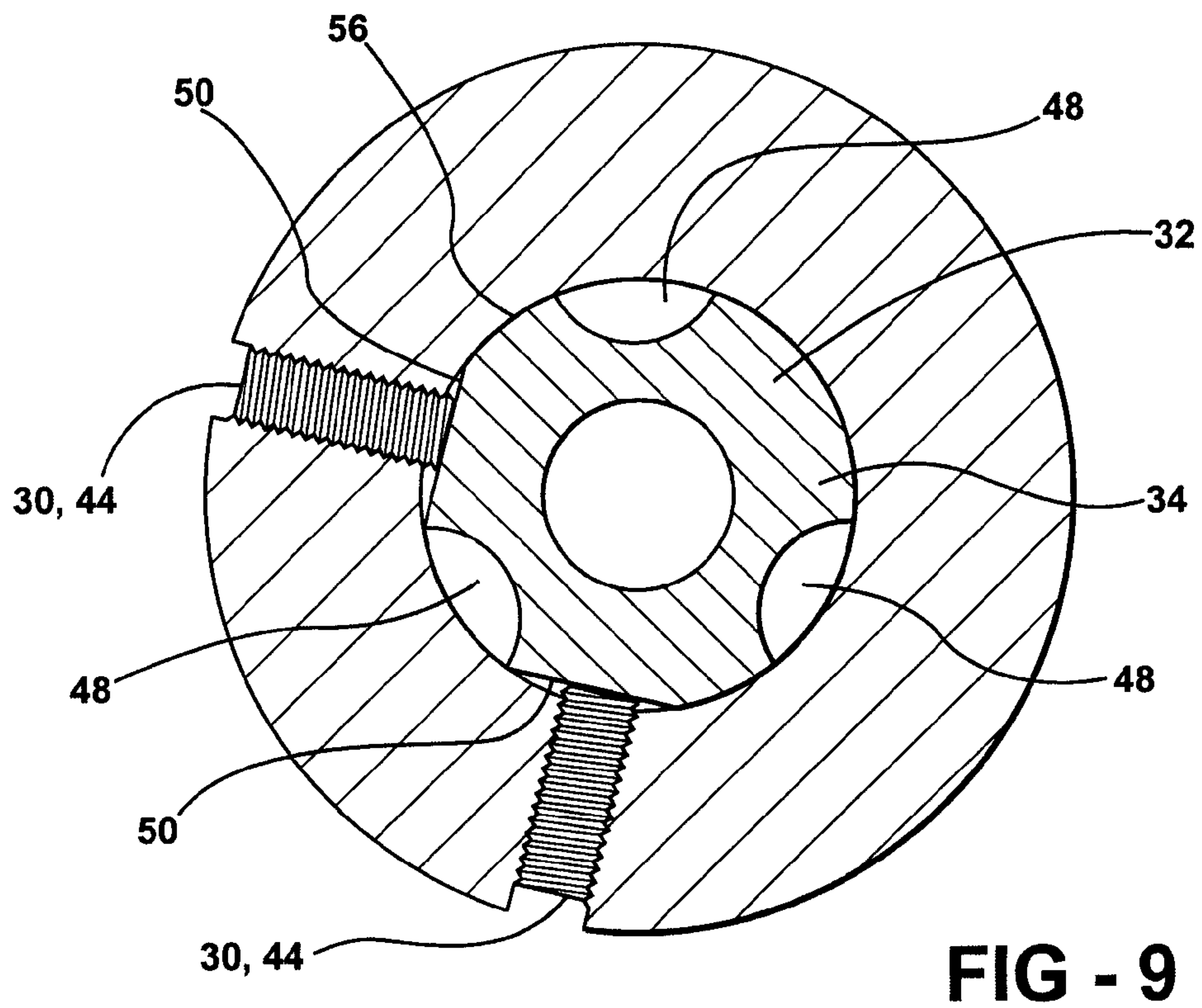
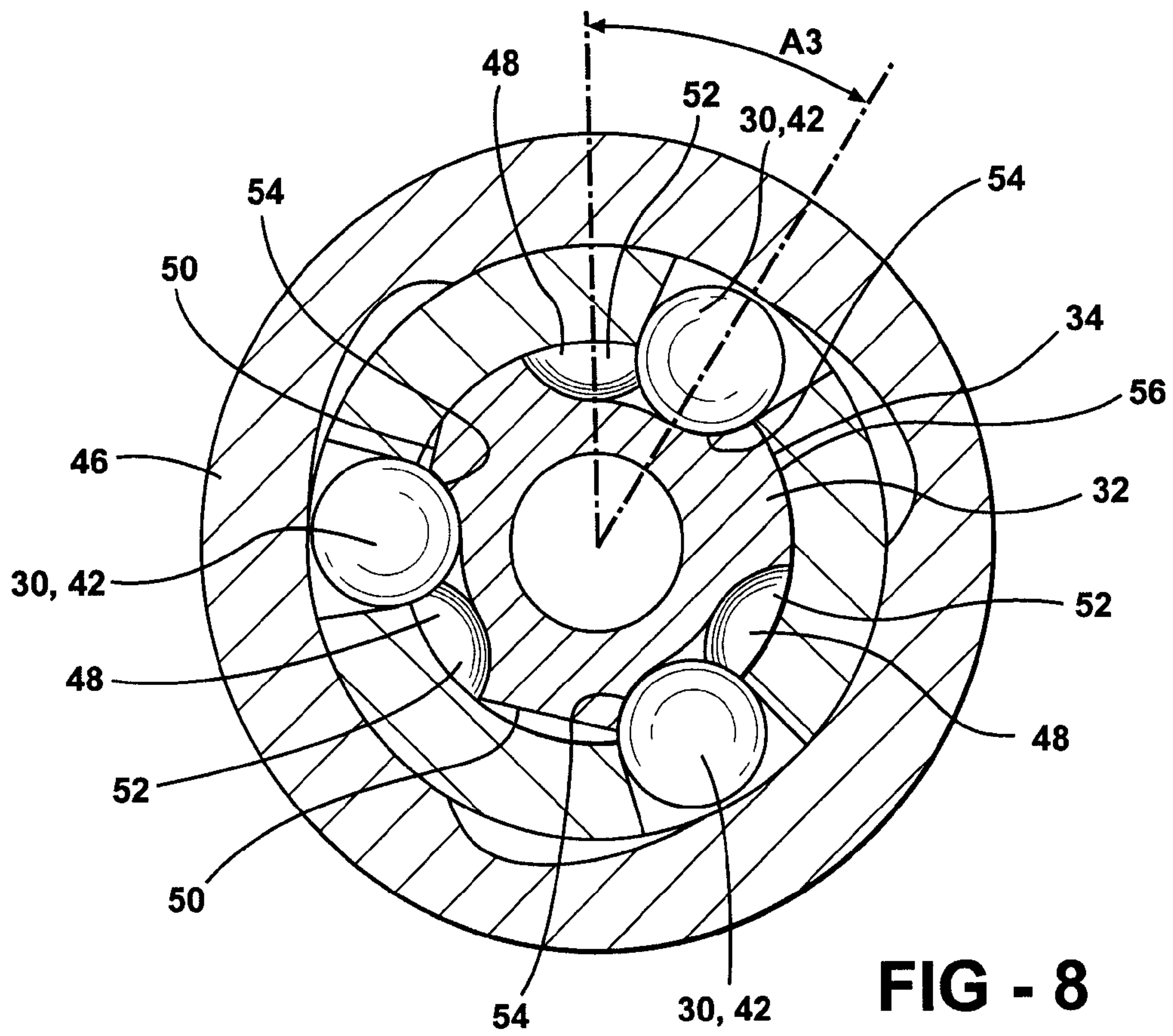


FIG - 7



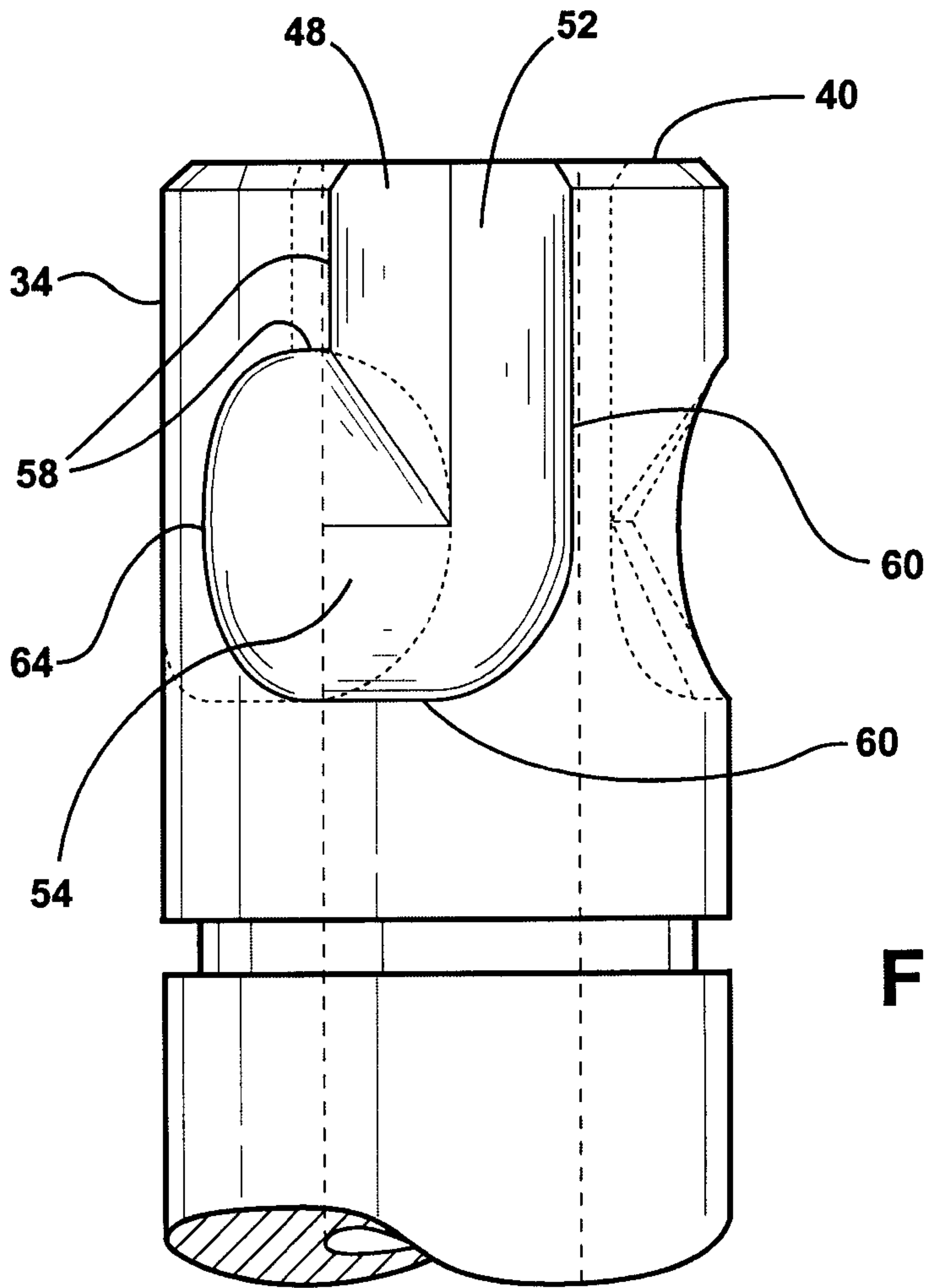


FIG - 10A

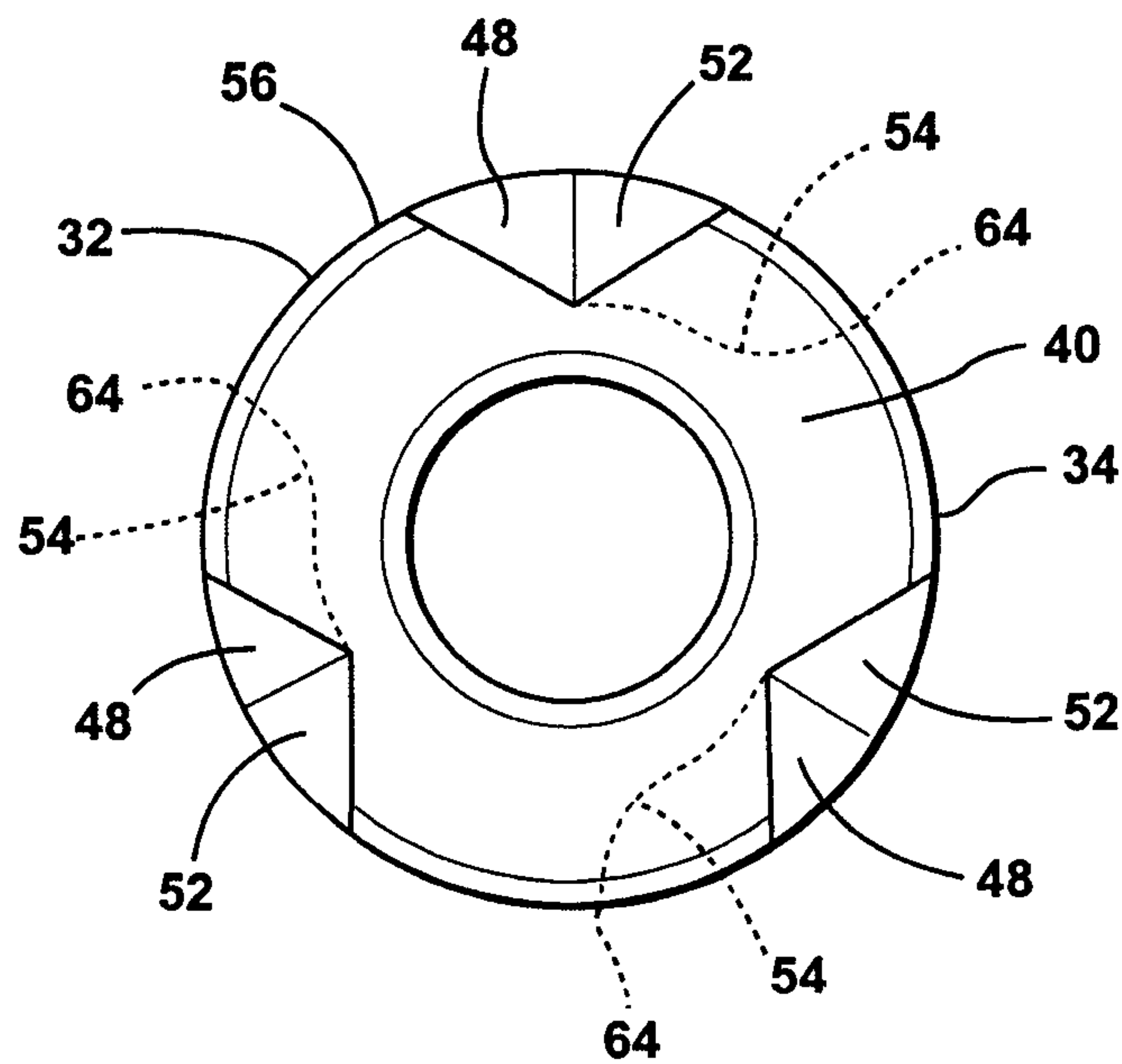


FIG - 10B

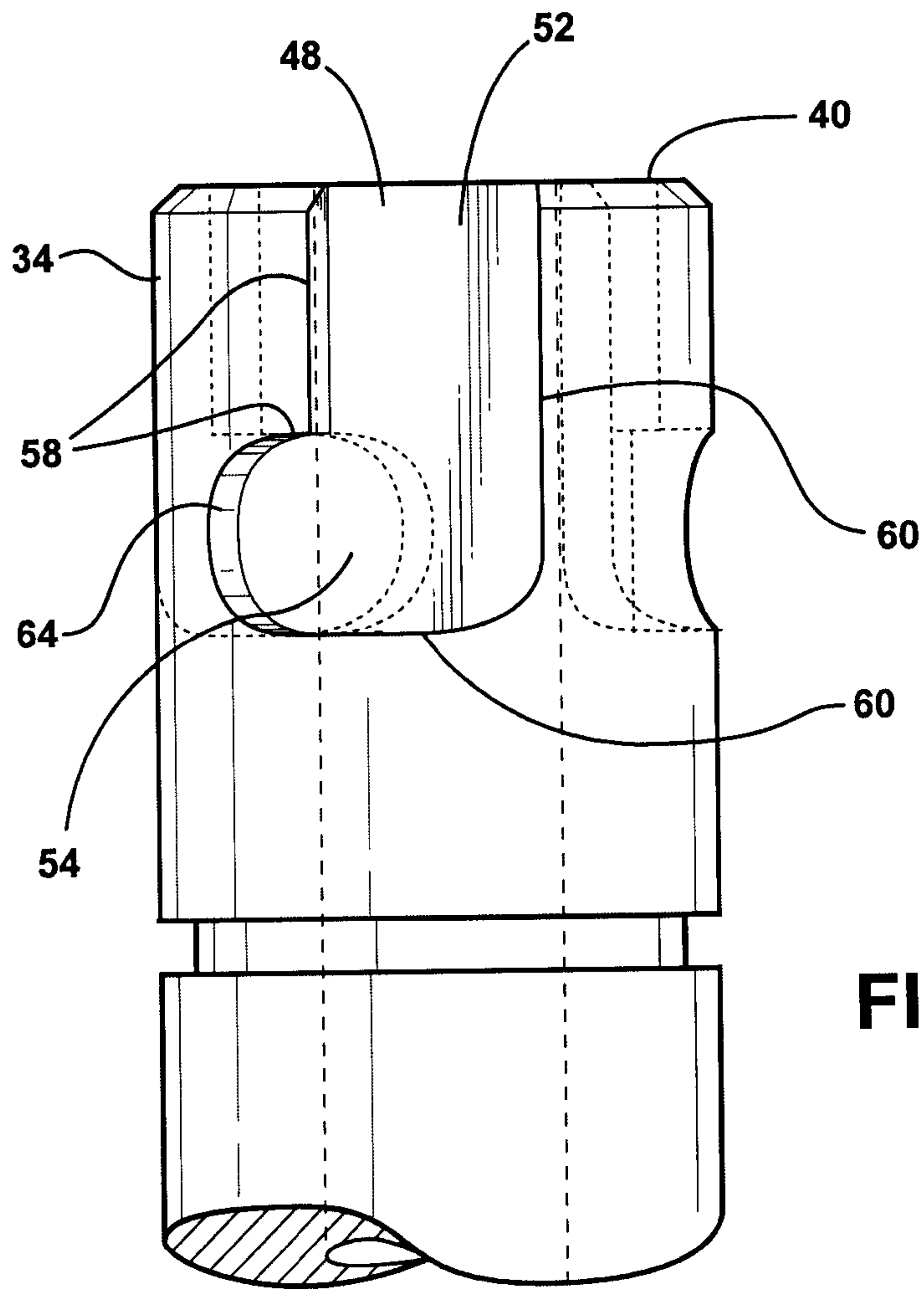


FIG - 11A

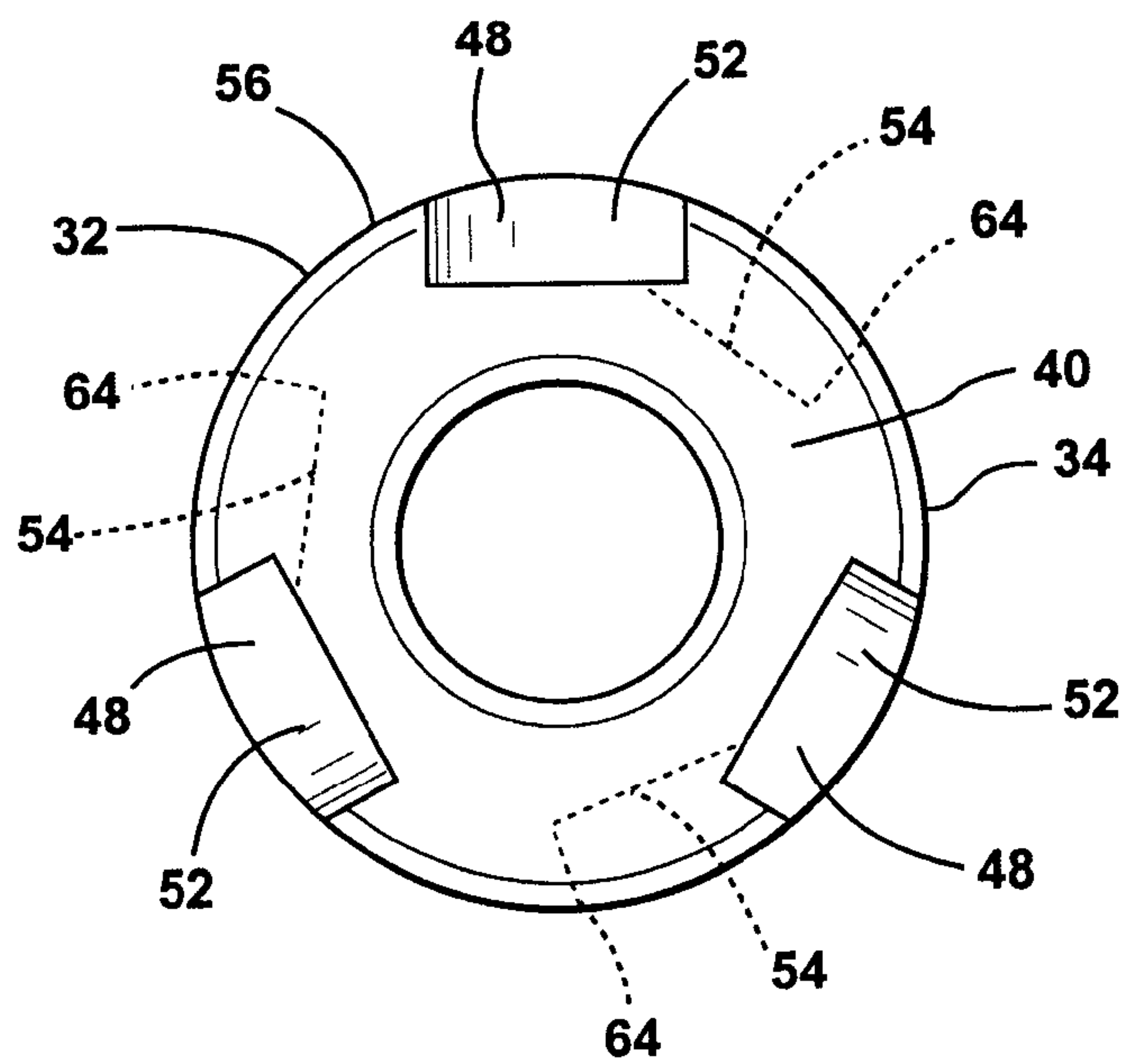


FIG - 11B

1

CUTTER FOR ENGAGEMENT WITH AN ARBOR

BACKGROUND OF THE INVENTION

1. Field of the Invention

The invention relates to a cutter, e.g., an annular cutter, for engagement with an arbor of a drill machine.

2. Description of the Related Art

Cutters, e.g., annular cutters, are known in the art for engagement with a drill machine for cutting holes in a workpiece. The drill machine includes a housing and a motor coupled to the housing. The arbor is coupled to the motor for rotation with the motor. The arbor defines a bore and the cutter is engaged to the arbor in the bore. A driving device is coupled to the arbor and extends into the bore for engaging the cutter. The driving device transmits rotation from the arbor to the cutter.

The driving device can be one of two types. Specifically, the driving device is either an extending member, e.g., a ball, extending into the bore of the arbor or a set screw threadedly extending through the arbor and into the bore.

When the driving device is the set screw, the cutter must define a flat such that the set screw can engage the flat. The set screw is rotated to threadedly advance and retreat the set screw into and out of the bore to engage and disengage the flat of the drill machine. It is known in the art to provide the cutter with a pair of flats arranged perpendicular to each other. The position of the pair of flats perpendicular to each other allows for stable engagement between the arbor and the cutter and allows for effective transmission of rotational motion from the arbor to the cutter.

When the driving device is the extending member, the cutter must define a retaining recess such that the extending member engages the cutter in the retaining recess. It is known in the art to provide a locking mechanism for moving the extending member radially within the bore. The locking mechanism moves the extending member radially outwardly to allow the cutter to be inserted into the bore and removed from the bore. The locking mechanism moves the extending member radially inwardly to engage the extending member with the retaining recess to lock the cutter in the bore so that the cutter can be operated.

There remains a need to provide a cutter that is interchangeable between arbors that include a pair of set screws and arbors that include the extending member. Specifically, there remains a need to provide a cutter that can be engaged with both the pair of set screws that extend perpendicularly to each other as well as the extending member.

SUMMARY OF THE INVENTION AND ADVANTAGES

The present invention is a cutter for driven engagement with an arbor having one of an extending member and a pair of set screws. The cutter comprises a shank portion having a central axis, a cutting portion extending from the shank portion along the central axis and a retaining recess formed in the shank portion for selectively receiving the extending member of the arbor when the arbor has the driving device. The cutter further comprises a pair of flats each formed in the shank portion for engaging the set screws when the arbor has the set screws with one of the flats extending in a first plane and with the other of the flats extending in a second plane perpendicular to the first plane.

Accordingly, the cutter can be used both with arbors that have an extending member and arbors that have a pair of set

2

screws. In other words, the cutter is interchangeable between such arbors. Further, the cutter advantageously has not only one flat but rather a pair of flats arranged perpendicular to each other. In addition, each of the pair of flats has sufficient surface area for engaging the set screws to more adequately transfer rotational movement from the arbor to the cutter.

BRIEF DESCRIPTION OF THE DRAWINGS

Other advantages of the present invention will be readily appreciated, as the same becomes better understood by reference to the following detailed description when considered in connection with the accompanying drawings wherein:

FIG. 1 is a cross-sectional view of a drill machine including a cutter disengaged from the drill machine and an extending member for engaging the cutter;

FIG. 2 is a cross-sectional view of the drill machine engaged with the extending member;

FIG. 3 is a cross-sectional view of another embodiment of the drill machine including a pair of set screws for engaging the cutter;

FIG. 4 is a side view of a portion of the cutter;

FIG. 5 is a top view of a shank portion of the cutter;

FIG. 6 is a cross-sectional view of the cutter through line 6 of FIG. 4;

FIG. 7 is a cross-sectional view of the cutter through line 7 of FIG. 4;

FIG. 8 is a cross-sectional view of the cutter through line 8 of FIG. 2;

FIG. 9 is a cross-sectional view of the cutter through line 9 of FIG. 3;

FIG. 10A is a side view of a portion of another embodiment of the cutter;

FIG. 10B is a top view of a portion of the embodiment of the cutter shown in FIG. 10A;

FIG. 11A is a side view of a portion of yet another embodiment of the cutter; and

FIG. 11B is a top view of a portion of the embodiment of the cutter shown in FIG. 11A.

DETAILED DESCRIPTION OF THE INVENTION

Referring to the Figures, wherein like numerals indicate like parts throughout the several views, a drill machine is generally shown at 20. As shown in FIGS. 1-3, the drill machine 20 includes a housing 22 and a motor 24 coupled to the housing 22. As appreciated by one skilled in the art, the housing 22 and the motor 24 are generically depicted in the Figures and the housing 22 and motor 24 can be of any type.

An arbor 26 is coupled to the motor 24 for rotation with the motor 24. The arbor 26 defines a bore 28 and a driving device 30 is coupled to the arbor 26 and extends into the bore 28. As will be discussed below and as best shown in FIGS. 2 and 3, a cutter 32 is partially inserted into the bore 28 and is coupled to the arbor 26 with the driving device 30. The cutter 32 partially extends from the bore 28 for cutting a workpiece (not shown), which is typically formed from a rigid material such as metal. The cutter 32 rotates relative to the housing 22 to cut a hole in the workpiece. For example, the cutter 32 is sized to cut holes sized between $\frac{7}{16}$ - $2\frac{3}{8}$ inches in diameter and 1-3 inches deep. It should be appreciated that the size of the holes can be any size without departing from the nature of the present invention.

The cutter 32 is more specifically referred to in industry as an annular cutter 32. The cutter 32 includes a shank portion 34 and a cutting portion 36. The cutter 32 extends along a central axis C and each of the shank portion 34 and the cutting portion

3

36 extend along the central axis C. The shank portion 34 extends along the central axis C between a first end 38 and a second end 40. The cutting portion 36 extends from the first end 38 of the shank portion 34. As best shown in FIGS. 2 and 3, the second end 40 of the shank portion 34 is inserted into the bore 28 of the arbor 26 and is coupled to the arbor 26 with the driving device 30 and the cutting portion 36 extends from the arbor 26.

The cutter 32 is typically formed from metal such as, for example, high speed steel. More specifically, for example, the high speed steel may be M2 steel or M42 steel. However, it should be appreciated that the cutter 32 can be formed from any rigid material known to one skilled in the art without departing from the nature of the present invention. The cutter 32 can include inserts, e.g., carbide inserts (not shown), along a tip of the cutting portion 36 to extend the life of the cutter 32.

The driving device 30 engages the cutter 32 to translate rotational motion from the arbor 26 to the cutter 32. The driving device 30 is further defined as one of an extending member 42 engaged with the arbor 26 in the bore 28, as best shown in FIGS. 2 and 8, and a pair of set screws 44, as best shown in FIGS. 3 and 9. As will be discussed below, in the configuration shown in FIGS. 2 and 8, the extending member 42 engages the cutter 32 and, in the configuration shown in FIGS. 3 and 9, the pair of set screws 44 engage the cutter 32. The extending member 42 shown in FIGS. 1 and 2 is further defined as a ball. However, it should be appreciated that the extending member 42 can be of any type that is engaged with the arbor 26 in the bore 28, such as, for example, a dowel pin. The extending member 42 is selectively moveable radially in the bore 28, i.e., toward and away from the shank portion 34. For example, in the configuration as best shown in FIGS. 1, 2, and 8, the arbor 26 includes a collar 46 that engages the extending member 42. The collar 46 is rotated to move the extending members 42 radially in the bore 28. Alternatively, the extending member 42 is fixed in position in the bore 28.

The shank portion 34 defines a retaining recess 48 and a pair of flats 50. The driving device 30 engages one of the retaining recess 48 and the pair of flats 50. In other words, the driving device 30 engages either the retaining recess 48 or at least one of the pair of flats 50. Specifically, when the driving device 30 is the extending member 42, as shown in FIGS. 2 and 8, the extending member 42 engages the retaining recess 48. When the driving device 30 is the pair of set screws 44, as shown in FIGS. 3 and 9, the pair of set screws 44 engages the pair of flats 50. In other words, the cutter 32 is interchangeable between arbors 26 that have driving members further defined as extending members 42, e.g., balls, and arbors 26 that have driving members further defined as set screws 44.

The retaining recess 48 can be further defined as a plurality of retaining recesses 48 spaced from each other circumferentially about the shank portion 34. In such a configuration, the driving device 30 is further defined as a plurality of extending members 42 spaced circumferentially in the bore 28 of the arbor 26 with each extending member 42 slideable along one of the retaining recesses 48, respectively. In the embodiment shown in Figures, the cutter 32 has three retaining recesses 48 and the arbor 26 shown in FIGS. 2 and 8 has three extending members 42 spaced about the bore 28 to engage each retaining recess 48. It should be appreciated that the cutter 32 can have any number of retaining recesses 48 and the arbor 26 can have a corresponding number of extending members 42 without departing from the nature of the present invention.

In an embodiment where the shank portion 34 has a plurality of retaining recesses 48, the retaining recesses 48 are typically evenly spaced circumferentially about the shank portion 34. For example, as best shown in FIG. 5, the three

4

retaining recesses 48 are circumferentially spaced 120 degrees apart from each other. However, it should be appreciated that the plurality of retaining recesses 48 can be spaced at any interval circumferentially about the shank portion 34.

As best shown in FIG. 4, the retaining recess 48 is further defined as an entry channel 52 and a retaining channel 54 extending from the entry channel 52, i.e., the shank portion 34 defines the entry channel 52 and the retaining channel 54. In the configuration shown in the Figures including the plurality of retaining recesses 48, each of the retaining recesses 48 include one of the entry channels 52 and one of the retaining channels 54 extending from the entry channel 52.

The entry channel 52 extends from the second end 40 of the shank portion 34 for guiding the driving device 30 of the arbor 26 during insertion of the shank portion 34 into the arbor 26. The retaining channel 54 selectively retains the driving device 30 of the arbor 26. Specifically, in the configuration shown in FIG. 2, the engagement between the retaining channel 54 and the extending member 42 vertically retains the cutter 32 in the arbor 26 and, as best shown in FIG. 8, rotation of the arbor 26 in the right direction, i.e., clockwise, drives the extending members 42 and the cutter 32 in the right direction.

In the configuration where the driving device 30 is further defined as at least one extending member 42, the extending members 42 are slideable along the entry and retaining channels 52, 54. For example, in the configuration where the driving device 30 is further defined as the ball engaged with the arbor 26, the ball is slideable along the entry and retaining channels 52, 54. The second end 40 of the shank portion 34 is inserted into the bore 28 of the arbor 26. The cutter 32 is rotated to align the extending members 42 with the entry channels 52 at the second end 40 of the shank portion 34. When the entry channels 52 are aligned with the extending members 42, the cutter 32 is moved into the bore 28 with the extending members 42 sliding along the entry channels 52. The extending members 42 are aligned with the retaining channels 54 when the extending members 42 are moved along the length of the entry channel 52. In the configuration shown in the Figures, when the extending members 42 are aligned with the retaining channels 54, the cutter 32 is rotated in the right direction, i.e., clockwise, to the position shown in FIG. 8. In such a configuration, the motor 24 rotates the arbor 26 in the right direction such that the extending members 42 drive the cutter 32 such that the cutter 32 is rotated in the right direction. It should be appreciated that the configuration of the shank portion 34 and the arbor 26 to accommodate the rotation of the motor 24 in the right direction is shown for exemplary purposes. Alternatively, the shank portion 34 and the arbor 26 may be configured to accommodate the rotation of the motor 24 in the left direction. In such a configuration, the shank portion 34 and the arbor 26 are mirror images of the configuration shown in FIG. 8 in which case the motor 24 rotates the arbor 26 in the left direction such that the extending members 42 drive the cutter 32 such that the cutter 32 is rotated in the left direction.

The method described above allows for quick engagement and disengagement of the cutter 32 and the arbor 26 because the only required movement is the axial translation of the cutter 32 in a direction along the central axis C and the rotational movement of the cutter 32 about the central axis C. The cutter 32 can typically be engaged and disengaged with the arbor 26 with the use of only one hand. Notably, the cutter 32 does not require an arbor 26 that has moving extending members 42 to move into and out of engagement with the retaining recess 48. In other words, the cutter 32 can be used in the arbor 26 that has extending members 42 fixed in posi-

5

tion in the bore 28 as well as the arbor 26 that has extending members 42 that selectively move into and out of the bore 28.

The shank portion 34 presents a cylindrical surface 56 extending circumferentially about the central axis C and the entry and retaining channels 52, 54 are defined in the cylindrical surface 56. Each of the entry and retaining channels 52, 54 has a first channel edge portion 58 and a second channel edge portion 60 spaced from each other with a trough portion 62 extending from the first channel edge portion 58 to the second channel edge portion 60. The first and second edge portions 58, 60 are further defined as boundaries between the cylindrical surface 56 and the trough portion 62, i.e., the first and second edge portions 58, 60 do not need to present angles between the cylindrical surface 56 and the trough portion 62 and can alternatively be rounded. Each trough portion 62 extends toward the central axis C between the first and second channel edge portions 58, 60. The trough portions 62 of each of the entry and retaining channels 52, 54 have a common configuration for slideably receiving the driving device 30.

The common configuration includes the depth of the trough portion 62, i.e., the trough portion 62 of each of the entry and retaining channels 52, 54 extend at a common depth D measured radially from the central axis C. The common depth D allows the extending member 42 to freely slide along the entry and retaining channels 52, 54 without interference with a bottom of the trough portion 62, i.e., without bottoming out. It should be appreciated that the entry and retaining channels can have differing depths without departing from the nature of the present invention. For example, the entry and/or retaining channels 52, 54 can present a ramp for selectively retaining the extending member 42 in the retaining channel 54.

Another common configuration includes the width of the trough portion 62, i.e., each of the entry and retaining channels 52, 54 has a width W measured between and transverse to the first and second channel edges with the width W of the entry channel 52 equal to the width W of the retaining channel 54. The common width W allows the extending member 42 to freely slide along the entry and retaining channels 52, 54 without interference with the first and second channel edge portions 58, 60. It should be appreciated that the entry and retaining channels 52, 54 can have different widths without departing from the nature of the present invention. Also, it should be appreciated that the widths of the entry and retaining channels 52, 54 can vary along the lengths of the entry and retaining channels 52, 54, respectively, without departing from the nature of the present invention.

Yet another common configuration includes the cross-sectional shape of the trough portion 62, i.e., each of the entry and retaining channels 52, 54 has a cross-sectional shape and the cross-sectional shape of the entry channel 52 is the same as the cross-sectional shape of the retaining channel 54. However, it should be appreciated that the cross-sectional shape of the entry channel 52 can be different than the cross-sectional shape of the retaining channel 54 without departing from the nature of the present invention. Typically, the driving device 30 is complimentary in configuration to the cross-sectional shape of the entry and retaining channels 52, 54, e.g., a cross-section of a portion of the extending member 42 that slides along the entry and retaining channels 52, 54 is the same as the cross-section of the entry and retaining channels 52, 54. In the configuration shown in FIGS. 1, 2, and 8, the cross-sectional shape is further defined as an arc and the driving devices 30, i.e., the balls, have a spherical shape. As best shown in FIG. 2, the cross-section of the portion of the extending member 42 that slides along the entry and retaining

6

channels 52, 54 is arced and thereby is complimentary in configuration to the cross-sectional shape of the entry and retaining channels 52, 54.

Alternatively, as shown in FIG. 10A-B, the cross-sectional shape of the trough portion 62 is V-shaped. In such a configuration, the cross-section of the portion of the extending member 42 that slides along the entry and retaining channels 52, 54 can be V-shaped. As shown in FIGS. 11A-B, the cross-sectional shape of the trough portion 62 is square-shaped. In such a configuration, the cross-section of the portion of the extending member 42 that slides along the entry and retaining channels 52, 54 can be squared. However, it should be appreciated that the extending members 42 can be further defined as balls having the spherical shape for use with both the V-shaped entry and retaining channels 52, 54 and the square-shaped entry and retaining channels 52, 54.

It should be appreciated that the cross-sectional shape of the trough portion 62 may be any shape without departing from the nature of the present invention. It should also be appreciated that while the driving device 30 is typically complimentary in configuration to the cross-sectional shape of the entry and retaining channels 52, 54, the cross-section of the portion of the driving device 30 that slides along the entry and retaining channels 52, 54 can be different than the cross-sectional shape of the trough portion 62 without departing from the nature of the present invention.

The entry channel 52 extends along the shank portion 34 in a first direction D1 and the retaining channel 54 extends along the shank portion 34 in a second direction D2 different than the first direction D1. Typically, the first direction D1 extends approximately perpendicular to the second direction D2. Specifically, as best shown in FIG. 4, the entry channel 52 extends axially along the shank portion 34. In other words, the entry channel 52 extends in parallel with the central axis C of the cutter 32. The retaining channel 54 extends circumferentially along the shank portion 34. However, it should be appreciated that the entry channel 52 can extend at an angle relative to the central axis C and the first direction D1 can extend at a non-perpendicular angle relative to the second direction D2 without departing from the nature of the present invention.

The entry portion extends along a longitudinal axis L and the retaining channel 54 extends at an angle A2 circumferentially about the central axis C from the longitudinal axis L to a retaining channel end 64. For example, the angle A2 is between 50 and 60 degrees. More specifically, the retaining channel 54 extends at approximately 55 degrees circumferentially about the central axis C from the longitudinal axis L to retaining channel end 64. For example, as best shown in FIG. 8, the extending member 42 is sized such that the cutter rotates at an angle A3 relative to the extending member. For example, the angle A3 is between 30 and 40 degrees. For example, in a configuration where the angle A3 is 35 degrees, when the second end 40 of the shank portion 34 is inserted into the bore 28 and the extending member 42 is aligned with the retaining channel 54, the cutter 32 is rotated approximately 35 degrees in the right direction to fully engage the cutter 32 with the arbor 26. It should be appreciated that angle A2 and angle A3 can be of any magnitude without departing from the nature of the present invention.

As best shown in FIGS. 5-7, one of the pair of flats 50 extends in a first plane P1 and the other of the pair of flats 50 extends in a second plane P2. As best shown in FIG. 6, the first and second planes P1, P2 extend at an angle A1 relative to each other. Preferably the angle A1 is 90 degrees, i.e., the second plane P2 extends perpendicular to the first plane P1. As best shown in FIG. 9, in the configuration where the extending member 42 is further defined as the pair of set

screws **44**, the arbor **26** defines set screw holes and the set screws **44** are threadedly engaged with the set screw holes. In the configuration where the angle **A1** is 90 degrees, the set screw holes are arranged approximately 90 degrees apart circumferentially and extend approximately perpendicularly relative to each other.

The position of the pair of flats **50** perpendicular to each other allows for stable engagement between the arbor **26** and the cutter **32**. Specifically, the pair of set screws **44** engaging the pair of perpendicular flats **50** forces the cutter **32** against the bore **28** of the arbor **26** and centers the cutter **32** in the arbor **26**. The position of the pair of flats **50** perpendicular to each other also allows for effective transmission of rotational motion from the arbor **26** to the cutter **32**. In other words, each of the set screws **44** engage the flats **50**, respectively, to transmit rotation of the arbor **26** to the cutter **32**.

In such a configuration, the second end **40** of the shank portion **34** is inserted into the bore **28** of the arbor **26**. The cutter **32** is rotated to align the pair of flats **50** with the pair of set screws **44**. When the pair of flats **50** are aligned with the pair of set screws **44**, the set screws **44** are threadedly advanced toward the cutter **32** to engage the pair of flats **50**, as shown in FIG. **9**. The motor **24** rotates the arbor **26** and the set screws **44** drive the cutter **32** such that the cutter **32** is rotated with the arbor **26**. It should be appreciated that the cutter **32** can also be used with an arbor **26** that has only one set screw **44** and in such a configuration the one set screw **44** is threadedly advanced to engage one of the pair of flats **50**.

As best shown in FIG. **6**, the pair of flats **50** are spaced from each other with the retaining recess **48** disposed between the pair of flats **50**. As best shown in FIG. **5**, the pair of flats **50** are recessed toward the central axis **C** from the cylindrical surface **56**. As best shown in FIG. **5**, the shank portion **34** has a reference plane **RP** extending through the central axis **C** and the longitudinal axis **L**. The first plane **P1** of one of the pair of flats **50** extends at an angle **A4** from the reference plane **RP** and the second plane **P2** of the other of the pair of flats **50** extends at an angle **A5** from the reference plane **RP**. In the embodiment shown in FIG. **5**, the first plane **P1** extends at approximately 45 degrees from the reference plane **RP**, i.e., the angle **A4** is 45 degrees, and the second plane **P2** extends at approximately 135 degrees from the reference plane **RP**, i.e., the angle **A5** is 135 degrees.

The drill machine **20** and the arbor **26** can include a lubrication system **66** as shown in FIGS. **1-3**. In such a configuration, the cutter **32** defines a hole extending along the central axis **C** and the cutter **32** slideably receives a pilot **68** in the hole. When the cutter **32** is contacted to the workpiece, the pilot **68** is forced upwardly to allow lubricant to flow from the arbor **26** toward the workpiece. It should be appreciated that the drill machine **20** and the arbor **26** can include any type of lubrication system **66** without departing from the nature of the present invention.

The invention has been described in an illustrative manner, and it is to be understood that the terminology which has been used is intended to be in the nature of words of description rather than of limitation. Obviously, many modifications and variations of the present invention are possible in light of the above teachings, and the invention may be practiced otherwise than as specifically described.

What is claimed is:

1. A drill machine comprising:
 - a housing;
 - a motor coupled to said housing;
 - an arbor defining a bore and coupled to said motor for rotation with said motor;

a driving device mounted to said arbor and extending into said bore; and

a cutter extending along a central axis and including a shank portion and a cutting portion each extending along said central axis;

said shank portion extending along said central axis between a first end and a second end with said cutting portion extending from said first end and with said shank portion terminating at said second end;

said shank portion defining a retaining recess and a pair of flats with one of said flats extending in a first plane and the other of said flats extending in a second plane perpendicular to said first plane and with said driving device engaging one of said retaining recess and said pair of flats;

wherein said retaining recess is further defined as an entry channel extending from said second end for guiding said driving device during insertion of said shank portion into said arbor and a retaining channel extending from said entry channel for selectively retaining said driving device; and

wherein each of said entry and retaining channels have a trough portion extending toward said central axis with said trough portions of each of said entry and retaining channels having a common configuration for slideably receiving said driving device.

2. The drill machine as set forth in claim **1** wherein said pair of flats are spaced from each other with said retaining recess disposed between said pair of flats.

3. The drill machine as set forth in claim **1** wherein said shank portion presents a cylindrical surface extending circumferentially about said central axis and wherein said pair of flats are recessed toward said central axis from said cylindrical surface.

4. A cutter for driven engagement with an arbor having one of an extending member and a pair of set screws, said cutter comprising:

a shank portion having a central axis and extending along said central axis between a first end and a second end, said cutter terminating at said second end;

a cutting portion extending from said first end of said shank portion along said central axis;

a retaining recess formed in said shank portion for selectively receiving the extending member of the arbor when the arbor has the driving device; and

a pair of flats each formed in said shank portion for engaging the set screws when the arbor has the set screws with one of said flats extending in a first plane and with the other of said flats extending in a second plane perpendicular to said first plane,

wherein said retaining recess is further defined as an entry channel extending from said second end for guiding said driving device during insertion of said shank portion into said arbor and a retaining channel extending from said entry channel for selectively retaining said driving device; and

wherein each of said entry and retaining channels have a trough portion extending toward said central axis with said trough portions of each of said entry and retaining channels having a common configuration for slideably receiving said driving device.

5. The drill machine as set forth in claim **1** wherein said entry channel extends along said shank portion in a first direction and said retaining channel extends along said shank portion in a second direction different than said first direction.

6. The drill machine as set forth in claim **5** wherein said second direction extends perpendicular to said first direction.

9

7. The drill machine as set forth in claim 1 wherein said entry channel extends axially along said shank portion and said retaining channel extends circumferentially along said shank portion.

8. The drill machine as set forth in claim 1 wherein said entry channel extends along a longitudinal axis and said retaining channel extends between 50 and 60 degrees circumferentially about said central axis from said longitudinal axis to a retaining channel end.

9. The drill machine as set forth in claim 8 wherein said shank portion has a reference plane extending through said central axis and said longitudinal axis and wherein said first plane of one of said pair of flats extends at 45 degrees from said reference plane and said second plane of the other of said pair of flats extends at 315 degrees from said reference plane.

10. The drill machine as set forth in claim 1 wherein said retaining recess is further defined as a plurality of retaining recesses spaced from each other circumferentially about said shank portion.

11. The drill machine as set forth in claim 1 wherein said driving device is further defined as one of an extending member engaged with said arbor in said bore and a pair of set screws.

12. The drill machine as set forth in claim 11 wherein said extending member is further defined as a ball.

13. The cutter as set forth in claim 4 wherein said retaining recess is further defined as a plurality of retaining recesses spaced from each other circumferentially about said shank portion.

10

14. The cutter as set forth in claim 4 wherein said flats are spaced from each other with said retaining recess disposed between said flats.

15. The cutter as set forth in claim 4 wherein said shank portion presents a cylindrical surface extending circumferentially about said central axis and wherein said flats are recessed toward said central axis from said cylindrical surface.

16. The cutter as set forth in claim 4 wherein said entry channel extends along said shank portion in a first direction and said retaining channel extends along said shank portion in a second direction different than said first direction.

17. The cutter as set forth in claim 16 wherein said second direction extends perpendicular to said first direction.

18. The cutter as set forth in claim 4 wherein said entry channel extends axially along said shank portion and said retaining channel extends circumferentially along said shank portion.

19. The cutter as set forth in claim 4 wherein said entry channel extends along a longitudinal axis and said retaining channel extends between 50 and 60 degrees circumferentially about said central axis from said longitudinal axis to a retaining channel end.

20. The cutter as set forth in claim 19 wherein said shank portion has a reference plane extending through said central axis and said longitudinal axis and wherein said first plane extends at 45 degrees from said reference plane and said second plane extends at 315 degrees from said reference plane.

* * * * *