

US007204474B2

# (12) United States Patent McGuire et al.

(10) Patent No.:

US 7,204,474 B2

(45) Date of Patent:

Apr. 17, 2007

(54)	HIGH-PRESSURE PLUG VALVE			
(75)	Inventors:	Bob McGuire, Oklahoma City, OK (US); L. Murray Dallas, Fairview, TX (US)		
(73)	Assignee:	Stinger Wellhead Protection, Inc., Allen, TX (US)		

Subject to any disclaimer, the term of this Notice: patent is extended or adjusted under 35

U.S.C. 154(b) by 31 days.

Appl. No.: 10/912,944

Aug. 6, 2004 (22)Filed:

#### **Prior Publication Data** (65)

Feb. 9, 2006 US 2006/0027779 A1

(51)	Int. Cl.	
	F16K 5/00	(2006.01)

- U.S. Cl. 251/309
- (58)See application file for complete search history.

#### (56)**References Cited**

#### U.S. PATENT DOCUMENTS

3,323,537	A	*	6/1967	Shafer
3,497,179	A	*	2/1970	Smyers, Jr 251/175
3,567,177	$\mathbf{A}$	*	3/1971	Mueller 251/188
3,750,430	A	*	8/1973	Crisa 70/1.5
3,768,512	A	*	10/1973	Lahaye
3,964,507	$\mathbf{A}$		6/1976	Jandrasi et al 137/375

4,132,388 A	*	1/1979	Billeter 251/175
4,257,447 A		3/1981	Clarkson
4,316,483 A		2/1982	Jandrasi
4,478,388 A	*	10/1984	George 251/309
5,123,439 A		6/1992	Powers 137/375
5,234,193 A	*	8/1993	Neal et al 251/175
5,577,708 A	*	11/1996	Pfannenschmidt 251/315.11
5,582,200 A		12/1996	Kimpel 137/375
5,586,749 A	*	12/1996	Conley et al 251/315.05
5,881,996 A	*	3/1999	Walsh et al 251/309
6,090,206 A	*	7/2000	Bang et al 118/715
6,446,664 B	1	9/2002	Parsons 137/375
6,675,825 B	1	1/2004	Reeves

#### OTHER PUBLICATIONS

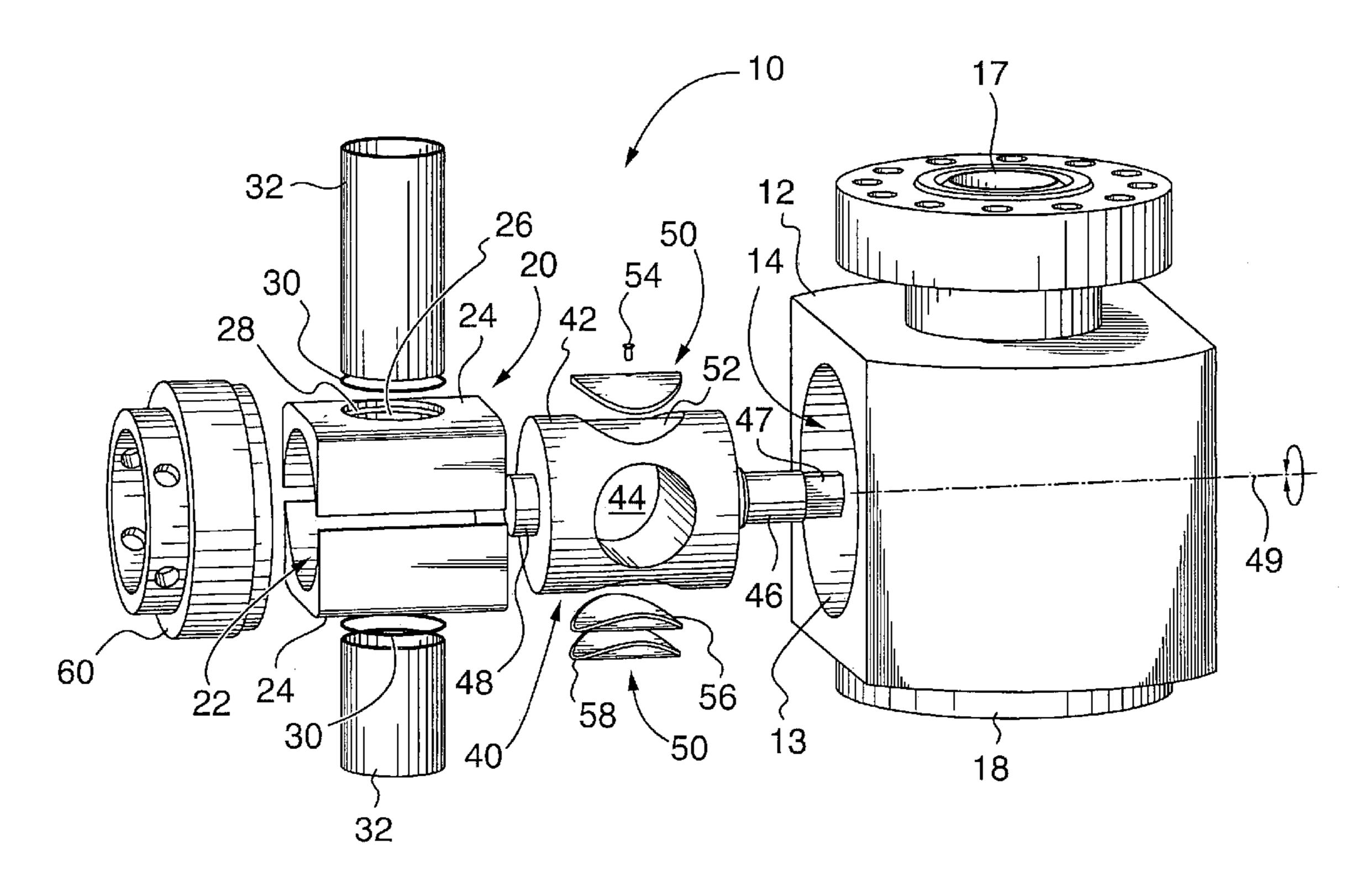
MSI Big Bore LT Brochure/Manual. Xomox Corporation, Process Valves & Actuators Brochure.

Primary Examiner—J. Casimer Jacyna (74) Attorney, Agent, or Firm—Nelson Mullins Riley & Scarborough, LLP

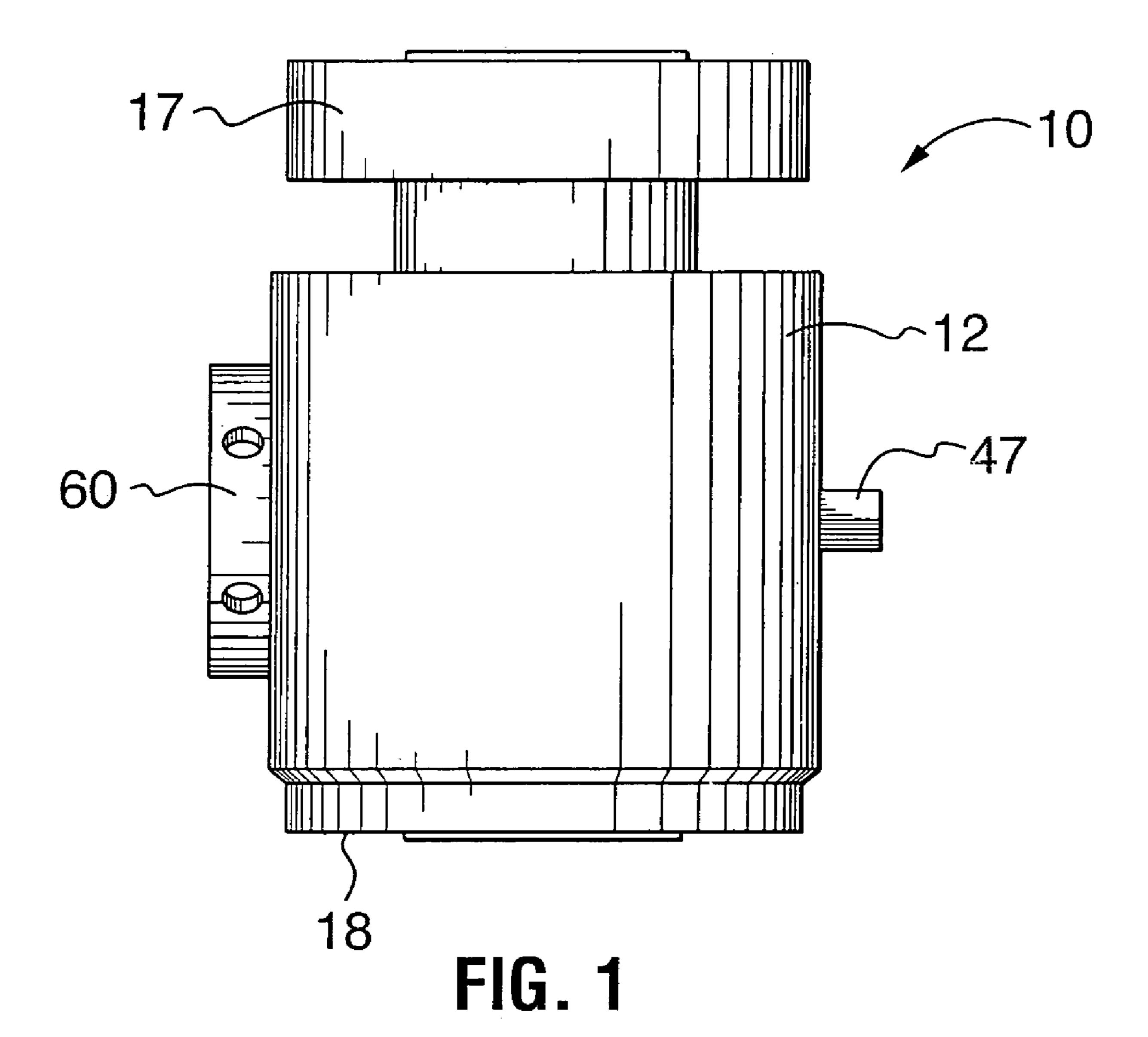
#### (57)**ABSTRACT**

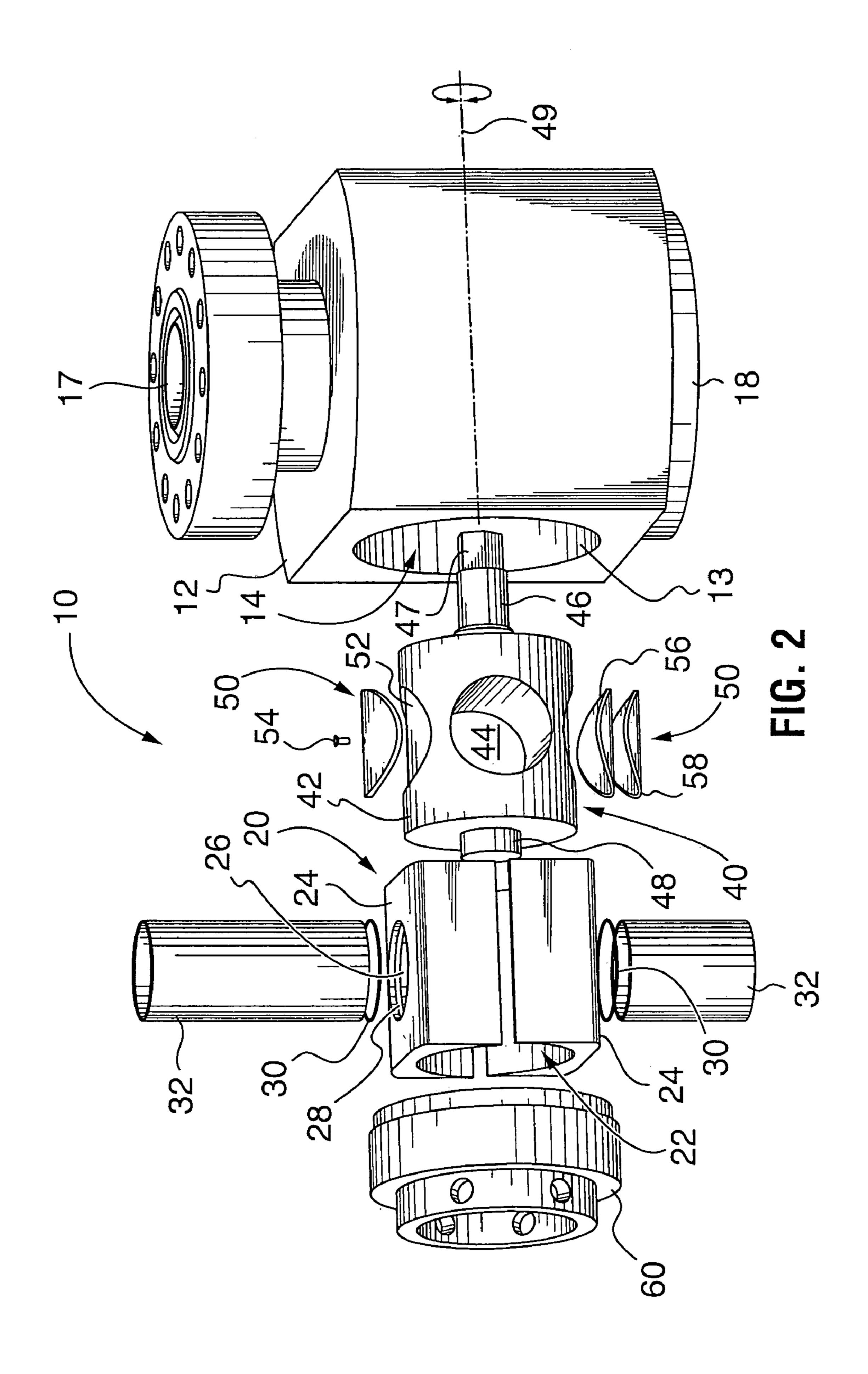
A plug valve has a valve body, a cage fitted within the valve body and a rotatable plug housed within a cylindrical cavity in the cage, for controlling fluid flow through the valve. The cage has first and second opposed flat side surfaces, each side surface has a port for receiving a replaceable flow-path insert in a fluid-tight seal. In one embodiment, the plug also includes at least one inset seal for sealing the at least one of the ports when the plug is in a closed position to inhibit leakage through the valve at low fluid pressures.

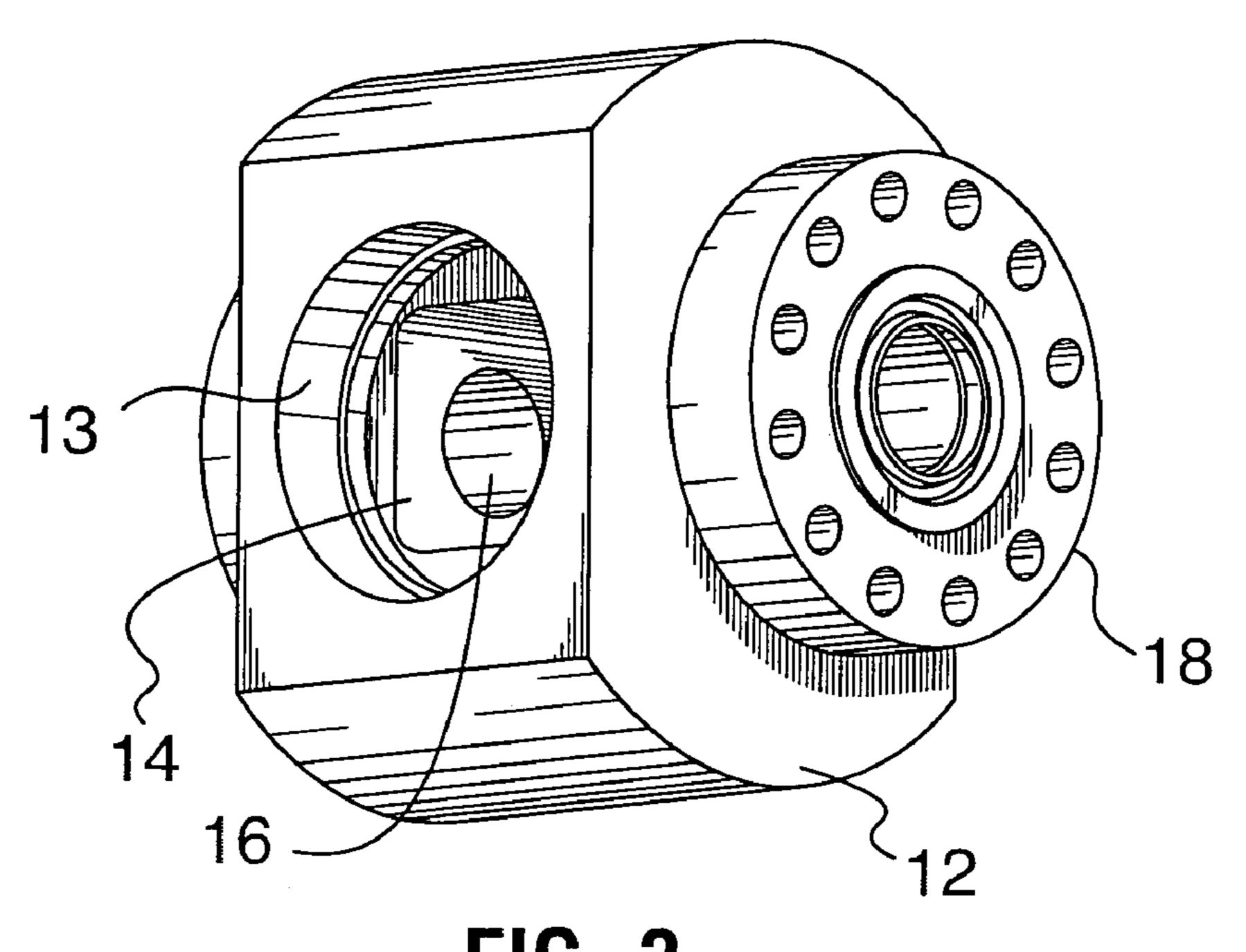
# 27 Claims, 3 Drawing Sheets

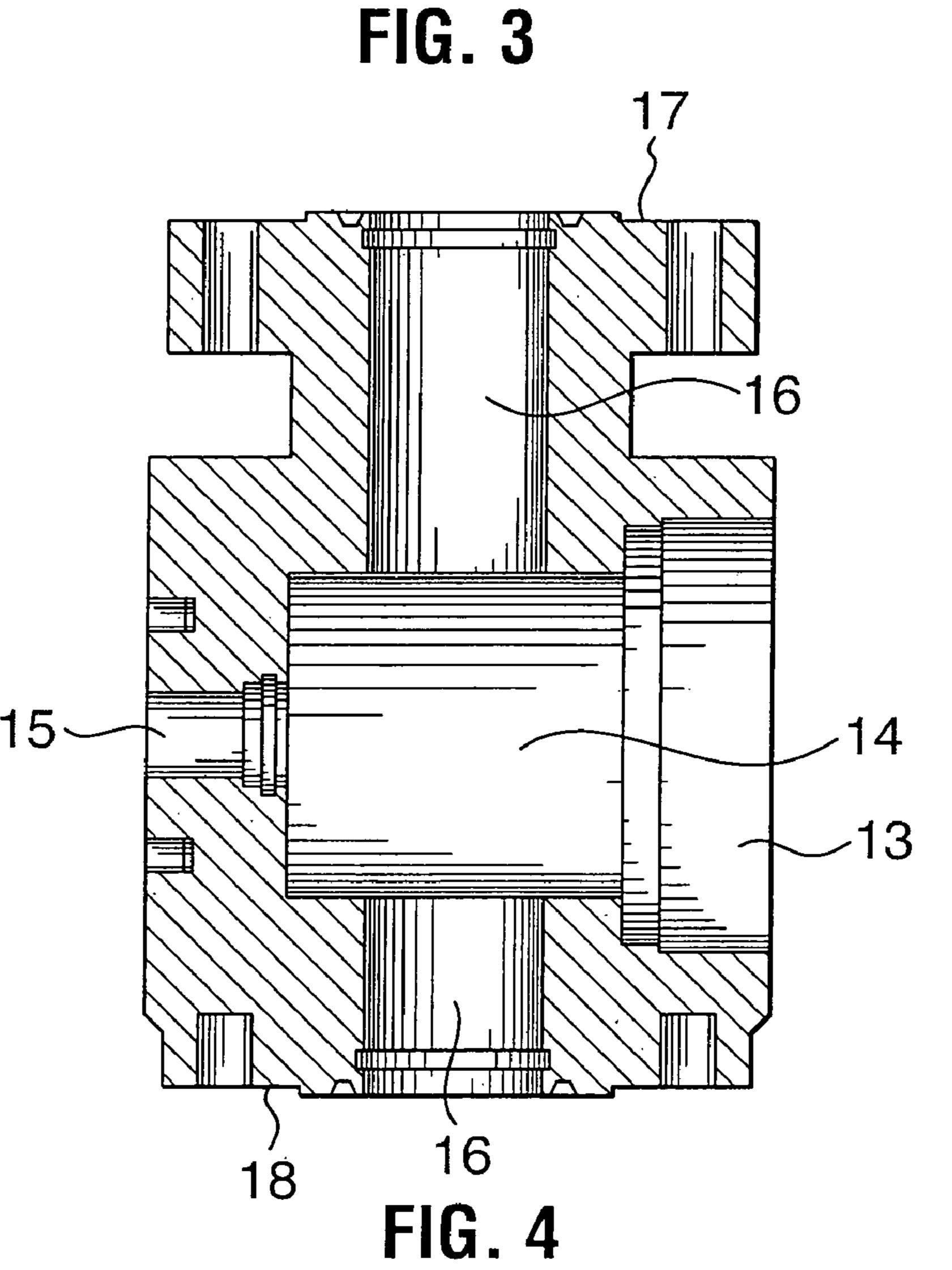


<sup>\*</sup> cited by examiner









10

# HIGH-PRESSURE PLUG VALVE

#### CROSS-REFERENCE TO RELATED APPLICATIONS

This is the first application filed for the present invention.

## MICROFICHE APPENDIX

Not Applicable.

#### TECHNICAL FIELD

The present invention relates in general to high-pressure valves for the oil and gas industry and, in particular, to plug 15 valves used in severe service applications in which flow path components, of the valve are subject to wear.

## BACKGROUND OF THE INVENTION

Plug valves are used in the oil and gas industry for heating and treating systems, cementing, fracturing and acidizing and other well stimulation applications in which high working pressures and abrasive and/or corrosive fluids are conducted at high flow rates.

Prior art plug valves are subject to wear. However, attempts have been made to permit at least some components in the flow path to be replaced. For example, U.S. Pat. No. 6,675,825 to Reeves et al., includes a valve body, an insert (or "cage") fitted non-rotatably within the valve body 30 and a plug rotatable within a cylindrical cavity in the cage between an open position, in which fluid flows through the valve, and a closed position, in which fluid flow is obstructed. In the open position, fluid flows through a bore through the plug. In the closed position, the bore in the plug is rotated perpendicular to the flow path and the plug blocks the flow path. When the cage and/or the plug become worn with use, one or both can be replaced without replacing the expensive valve body.

In use, a plug valve is frequently subjected to high 40 of the expensive plug valve body. pressures, corrosive fluids and abrasive proppants (such as sand, resin-coated sand or sintered bauxite) which tend to erode the components of the plug valve that form the flow path through the valve. As is well known in the art, component wear causes close-fitting components to loose their 45 initial shape, thereby rendering the plug valve more prone to leaking. In order to mitigate the effects of wear, plug valves used in severe service conditions require regular maintenance and refurbishment. The non-replaceable components of plug valves are typically refurbished by welding new 50 metal in the flow path, and then machining the flow path components back to their original tolerances. Not only is refurbishment by welding and machining time-consuming and expensive, but the mechanical and thermal properties of the weld are inferior to those of the steel used to make the 55 an elastomeric material bonded to a metal backing. plug valve. Furthermore, for sour service, where welds are exposed to H<sub>2</sub>S, the welds are susceptible to sulphide stress corrosion cracking (SSCC).

Consequently, there is a need for a plug valve with improved erosion resistance that is more quickly and easily 60 refurbished to original specifications.

A further common problem with high pressure plug valves is low pressure containment. Although plug valves generally provide a very reliable seal at high fluid pressures because in the closed position the high fluid pressures force 65 the plug into a metal-to-metal sealing contact with the plug cage. However, when fluid pressures are low, especially with

valves that have seen some severe service, the low pressure fluids may leak around the plug. This problem is most often experienced when setting up for a well stimulation treatment, or after flow back of the well before the pressure isolation equipment is removed from the wellhead. Leakage around the plug valve can permit hydrocarbons to escape to atmosphere, which is very undesirable.

There therefore also exists a need for a plug valve that provides a reliable seal at low fluid pressures.

#### SUMMARY OF THE INVENTION

It is therefore an object of the invention to provide a high-pressure plug valve with improved wear resistance.

It is another object of the invention to provide a plug valve that is quickly and easily refurbished to original specifications.

It is yet another object of the invention to provide a plug valve that provides a reliable fluid seal under low fluid 20 pressure conditions.

In accordance with a first aspect of the invention, a plug valve includes a valve body and a replaceable cage fitted within the valve body, the cage defining a cylindrical cavity for receiving a plug for controlling fluid flow through the valve, the plug being rotatable between an open position in which fluid flows through the valve, and a closed position in which fluid flow is obstructed, and the cage has first and second opposed flat side surfaces, each surface having a port for receiving an end of a flow-path insert that defines a flow path through one side of the valve.

In one embodiment, each port includes a circumferential groove for receiving an annular sealing element to provide a fluid seal between the flow path insert and the cage.

Because the cage has flat side surfaces, the seals can be provided between the flow-path inserts and the ports. Furthermore, when the flow-path inserts and/or the cage has worn beyond an acceptable tolerance, the flow path inserts and the cage are easily and quickly replaced, thus reducing maintenance costs and significantly improving the longevity

In accordance with a second aspect of the invention, a plug valve includes a valve body with first and second opposed ports in fluid communications with a flow path through the valve, the first and second ports communicating with a cylindrical cavity for receiving a plug for controlling fluid flow through the valve, the plug being rotatable between an open position, in which fluid flows through the valve, and a closed position, in which fluid flow is obstructed, and the plug comprises at least one inset seal for sealing at least one of the ports when the plug is in the closed position, the at least one inset seal being resiliently biased against at least one port when the plug is in a closed position to provide a fluid-tight seal at low fluid pressures.

In one embodiment, the at least one inset seal comprises

The inset seal provides low-pressure sealing of the port, thereby inhibiting leakage at low fluid pressures which complement a metal-to-metal seal that forms between the plug and the cage at high fluid pressures. The plug valve in accordance with the second aspect of the invention thus inhibits leakage throughout a full range of operating pressures.

In accordance with a third aspect of the invention, a plug valve includes a valve body and a replaceable cage fitted within the valve body, the cage having first and second opposed ports for receiving flow-path inserts defining a flow path through the valve, the cage further defining a cylindri-

cal space for receiving a plug for controlling fluid flow through the valve, the plug being rotatable between an open position, in which fluid flows through the valve, and a closed position, in which fluid flow is obstructed; wherein the plug comprises a pair of inset seals for sealing the ports when the 5 plug is in the closed position, the inset seals being resiliently biased against the ports when the plug is in a closed position to provide a fluid-tight seal at low fluid pressures, and a closed position, in which fluid flow is obstructed.

In one embodiment, an annular sealing element provides 10 a fluid-tight seal between each flow-path insert and the respective port.

## BRIEF DESCRIPTION OF THE DRAWINGS

Further features and advantages of the present invention will become apparent from the following detailed description, taken in combination with the appended drawings, in which:

- FIG. 1 is a side elevational view of a plug valve in accordance with the invention;
- FIG. 2 is an exploded, perspective view of a plug valve in accordance with an embodiment of the invention;
- FIG. 3 is a perspective view of a valve body for use with the plug valve shown in FIG. 1; and
- FIG. 4 is a cross-sectional view of the valve body shown in FIG. 2.

It will be noted that throughout the appended drawings, like features are identified by like reference numerals.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

In general, and as will be explained in detail below, a plug 35 valve in accordance with the invention includes a valve body, a cage fitted within the valve body and a rotatable plug housed within a cylindrical cavity in the cage. The plug can be rotated between an open position, in which fluid flows through the valve, and a closed position, in which fluid flow 40 is obstructed. The cage has first and second opposed flat side surfaces, each side surface having a port for receiving a replaceable flow-path insert. In one embodiment, an annular sealing element, such as an elastomeric O-ring, provides a fluid-tight seal between the flow-path insert and the port. In 45 another embodiment, the plug also includes at least one inset seal shaped to be substantially flush with the cylindrical surface of the plug. The inset seal has an outer surface of an elastomeric material for sealing at least one of the ports when the plug is in the closed position, thereby inhibiting 50 leakage through the valve at low fluid pressures.

FIG. 1 is a side elevational view of a plug valve, generally indicated by the reference numeral 10, in accordance with an embodiment the invention. The plug valve includes a plug body 12, which is the most expensive component of the plug 55 valve. The plug valve further includes a flange 17 for connecting the plug valve to other wellhead or flow control components and a stud pad 18. As is understood by those skilled in the art, either of the flange 17 and the stud pad 18 can be replaced with the other, so that the plug valve 10 60 NACE Trim is a stainless steel specified in the standards of includes either two flanges 17 or two stud pads 18. As will be explained below in further detail with reference to FIGS. 2–4, plug valve 60 further includes a hexagonal lug 47 to which a valve lever or valve wheel is connected in a manner well known in the art to move a plug of the valve from an 65 open to a closed position. Plug valve 10 further includes a cap 60 which is removed to provide access to internal

components of the plug valve 10, as will likewise be explained in more detail with reference to FIGS. 2–4.

FIG. 2 is an exploded, perspective view of the plug valve 10 in accordance with an embodiment of the invention. The plug valve 10 includes a valve body 12 and a cage 20 (also referred to as a plug insert) which is non-rotatably received within the valve body 12. A plug 40 is housed within the cage 20 and is rotatable between an open position, in which fluid flows through the valve, and a closed position, in which fluid flow is obstructed.

The cage 20 is received in a socket 14, shown in FIGS. 3 and 4, that is machined in the valve body 12 using a milling machine, for example. The socket 14 has flat, parallel side surfaces.

As shown in FIGS. 3 and 4, the valve body 12 also has a shallow, large-diameter cap bore 13 for receiving the cap 60. FIG. 3 also shows that the valve body is machined to have a stem bore 15 for accommodating a stem 46 of a valve plug 40 (FIG. 2), also described below. In addition, flow-path bores 16 are machined into the valve body 12 for receiving flow-path inserts 32 (FIG. 2), as will likewise be described below.

In one embodiment, the valve body 12 is machined from AISI 4130 or 4140 steel having a Rockwell Hardness of 25 about 18–32.

As shown in FIG. 2, the cage 20 is a generally rectangular box-shaped insert which defines a cylindrical cavity 22 for receiving a generally cylindrical plug 40. The cage 20 can be machined as two symmetrical pieces (as shown) or as a 30 single integral piece. In one embodiment, the cage 20 is machined from ductile iron having a yield strength of about 40–50 kpsi (275–345 MPa). In one embodiment, the iron is nitrite-hardened to have surface hardness of about Rockwell Hardness **60**.

The cage 20 has first and second side surfaces 24 that are substantially flat. For the purposes of this specification, the qualification that the first and second side surfaces are substantially flat means that the exterior surfaces are substantially flat, not the interior surfaces which are curved to define the cylindrical cavity 22 for receiving the cylindrical plug **40**.

The first and second side surfaces 24 are parallel and located on opposite sides of the cylindrical cavity 22. The first surface includes a circular port 26 which is aligned with a second circular port 26 in the second side surface 24. In one embodiment, each port 26 includes a circumferential groove 28 for receiving an annular sealing element 30, such as an O-ring. Each port **26** is a circular aperture adapted to receive an end of a replaceable flow-path insert 32. The flow-path inserts 32 are inserted into the flow-path bores 16 in the valve body 12 (see FIGS. 3 and 4). The annular sealing element 30, which is received in the circumferential groove 28 in each port 26, thus provides a fluid-tight seal between the cage 20 and the end of the flow-path insert 32. In one embodiment, the flow-path inserts are machined from AISI 4340 heat-treated steel having a Rockwell Hardness of about 50–52. Alternatively, for sour service, the flow-path inserts may be machined from HH 1150 NACE Trim stainless steel having a Rockwell Hardness of about 35. The HH 1150 the National Association of Corrosion Engineers and is known in the art to provide superior corrosion resistance for use in certain sour service environments.

As shown in FIG. 2, the plug 40 is a generally cylindrical body 42 through which a transverse bore 44 is machined. The transverse bore 44 provides a flow passage through which fluid may flow when the plug is in the open position.

5

The plug 40 has a stem 46 that protrudes from a first end of the plug and which is housed within the stem bore 15 (FIG. 4) in the valve body. The stem 46 includes a hexagonal lug 47 to which is mounted a valve lever or a valve wheel for rotating the plug from the open position to the closed 5 position, or vice versa. The plug 40 includes a stubby, cylindrical axial stud 48 that protrudes from an opposite end of the plug, for aligning the plug within the cage 22 and the valve body 12. In one embodiment, the plug is machined from nitrite-hardened HH 1150 NACE Trim stainless steel.

In one embodiment, the plug valve 10 further includes at least one inset seal 50 which fits within a machined recess 52 in an outer surface of the body 42 of the plug 40. In one embodiment, a machined fastener such as a screw 54 is used to fasten the inset seal 50 to the plug 40. The inset seal 50 15 has the same curvature as the plug 40 so that the inset seal 50 fits flush with the outer surface of the plug. In one embodiment, the inset seal 50 is replaceable and includes an elastomeric material 56 bonded to a metal backing 58. As will be understood by those skilled in the art, the insert seal 20 50 may likewise be an elastomeric material bonded directly to the machined recess 57. In one embodiment, a pair of inset seals 50 is provided, as shown in FIG. 2, so that low pressure sealing is provided regardless of an orientation of the plug valve 10.

When the plug is in the closed position, the inset seal 50 is resiliently biased against the port 26 of the cage 20, thereby providing a fluid-tight seal at low fluid pressures. In one embodiment, the elastomeric material 56 is polyure-thane or nitrile rubber with a durometer of 70–100. The 30 metal backing 58 is made of stainless steel, aluminum, bronze or brass, for example. When the inset seal 50 becomes worn, it can be removed (by removing the screw 54) and replaced with a new inset seal 50, thus prolonging the low-pressure service life of the plug valve 10.

As described above, the valve body 12 has a cap bore 13 that threadedly secures the cap 60. The cap 60 is threadedly secured to the valve body 12 to retain the cage 20 and plug 40 within the socket 14 of the valve body 12. In one embodiment, the cap 60 is machined from AISI 4140 40 heat-treated steel.

In operation, the plug valve 10 can be opened and closed by rotating the plug 40 through ninety degrees by applying torque to the hexagonal lug 47 using a wheel, lever or other such valve actuator. As was described above, the plug 40 can 45 be rotated from an open position, in which fluid may flow through the valve, to a closed position, in which fluid flow is obstructed. The plug 40, stem 46 and stud 48 define an axis of rotation 49 within the cylindrical cavity 22 in the cage 20.

When the plug valve 10 is opened and closed, the transverse bore 44 and inset seals 50 rotate about the axis of rotation. In the open position, the transverse bore 44 aligns with the ports 26 and flow-path inserts 32, thus providing the flow path through the valve. In the closed position, the transverse bore 44 is perpendicular to the flow path, thereby obstructing fluid flow. In the closed position, the inset seals 50 align with the ports 26 and flow-path inserts 32 to seal the ports 26 of the cage 20. In the closed position, the elastomeric inset seals 50 are resiliently biased against the ports 60 backing.

26, thereby inhibiting leakage at low fluid pressures.

Complementing the low-pressure inset seals 50 is a metalto-metal seal that forms between the plug 40 and the cage 20 at high fluid pressures. The plug valve 10 thus inhibits leakage throughout a full range of operating pressures.

The plug valve 10 can be efficiently serviced by replacing worn out components rather than refurbishing the valve by

6

welding and re-machining. The replaceable components of the plug valve 10 are the plug 40, the inset seals 50 and associated fasteners 54, the cage 20, and the flow-path inserts 32 with associated annular sealing elements 30. By disassembling the plug valve 10, these components can be quickly and efficiently replaced to refurbish the plug valve 10 to its original specifications. Since the expensive valve body 12 is untouched by abrasive fluids during service, the valve body can continue to be used for many years. The plug valve 10 therefore reduces maintenance costs because plug valve 10 is quickly refurbished by workmen without sophisticated welding and machining skills. The plug valve 10 also reduces overhead because valves can be quickly refurbished, so fewer valves are required.

Persons of ordinary skill in the art will appreciate, in light of this specification, that minor variations may be made to the components of the plug valve without departing from the sprit and scope of the invention. The embodiments of the invention described above are therefore intended to be exemplary only and the scope of the invention is limited only by the scope of the appended claims.

We claim:

- 1. A plug valve comprising:
- a valve body;
- a replaceable cage received within the valve body, the replaceable cage defining a cylindrical cavity for receiving a plug for controlling fluid flow through the plug valve, the plug being rotatable between an open position, in which fluid flows through the plug valve, and a closed position, in which fluid flow is obstructed, the cage having first and second opposed flat side surfaces, each side surface including a port; and
- a pair of flow-path inserts respectively having an inner end and an outer end, the inner end of each flow path insert being received in one of the ports, the flow-path inserts defining a flow path through the plug valve that is obstructed by rotation of the plug to the closed position.
- 2. The plug valve as claimed in claim 1 wherein the cage comprises four flat side surfaces, thereby defining a generally rectangular box-shaped insert.
- 3. The plug valve as claimed in claim 1 wherein the cage is a unitary structure.
- 4. The plug valve as claimed in claim 1 wherein the cage is a two-piece structure, each piece including a one of the ports.
- 5. The plug valve as claimed in claim 1 wherein each port comprises a circumferential groove for receiving an annular sealing element to provide a fluid seal around the inner end of the flow-path insert received in the port.
  - 6. The plug valve as claimed in claim 1 wherein the plug comprises a pair of inset seals for sealing the ports when the plug is in the closed position, the inset seals being resiliently biased against the ports to provide a fluid-tight seal at low fluid pressures.
  - 7. The plug valve as claimed in claim 6 wherein each inset seal is substantially flush with an outer surface of the plug.
  - 8. The plug valve as claimed in claim 7 wherein the inset seal comprises an elastomeric material bonded to a metal backing.
  - 9. The plug valve as claimed in claim 8 wherein the metal backing comprises a metal disk to which the elastomeric material is bonded.
- 10. The plug valve as claimed in claim 8 wherein the metal backing comprises a machined surface of the plug and the elastomeric material is bonded directly to the machined surface.

- 11. The plug valve as claimed in claim 8 wherein the elastomeric material is one of a polyurethane and a nitrile rubber.
  - 12. A plug valve comprising:
  - a valve body; and
  - a replaceable cage non-rotatably received within the valve body, the replacable cage having first and second opposed flat, parallel side surfaces, each side surface including a port for removably receiving a replaceable flow-path insert that collectively define a flow path 10 through the valve body, the replacable cage further defining a cylindrical cavity for receiving a plug for controlling fluid flow through the valve body, the plug being rotatable between an open position, in which fluid flows through the valve body, and a closed posi- 15 pressures, comprising: tion, in which fluid flow is obstructed.
- 13. The plug valve as claimed in claim 12 wherein each replaceable flow-path insert is tubular and has a planar end received in a respective one of the first and second ports.
- 14. The plug valve as claimed in claim 12 further com- 20 prising an annular sealing element for providing a fluid-tight seal between each flow-path insert and the respective first and second ports.
- 15. The plug valve as claimed in claim 14 wherein the annular sealing elements comprise O-rings.
- 16. The plug valve as claimed in claim 12 wherein the replacable cage is made of a ductile metal having a hardened surface.
- 17. The plug valve as claimed in claim 12 wherein the plug comprises at least one inset seal for sealing at least one 30 of the first and second opposed ports when the plug is in the closed position, the at least one inset seal being resiliently biased against the one of the first and second opposed ports to provide a fluid-tight seal at low fluid pressures when the plug is in the closed position.
- **18**. The plug valve as claimed in claim **17** wherein the at least one inset seal comprises a pair of inset seals for respectively sealing each of the first and second opposed ports.
- **19**. The plug valve as claimed in claim **17** wherein the at 40 least one inset seal is substantially flush with an outer surface of the plug.
- 20. The plug valve as claimed in claim 17 wherein the at least one inset seal comprises an elastomeric material bonded to a metal backing.

8

- 21. The plug valve as claimed in claim 20 wherein the metal backing comprises a metal disk to which the elastomeric material is bonded.
- 22. The plug valve as claimed in claim 21 wherein the at 5 least one inset seal is secured to the plug by at least one mechanical fastener.
  - 23. The plug valve as claimed in claim 20 wherein the metal backing comprises a machined surface of the plug and the elastomeric material is bonded directly to the machined surface.
  - 24. The plug valve as claimed in claim 20 wherein the elastomeric material is one of polyurethane and nitrile rubber.
  - 25. A plug valve that provides a reliable seal at low fluid
    - a valve body;
    - a replaceable cage received within the valve body, the replacable cage defining a cylindrical cavity for receiving a plug for controlling fluid flow through the plug valve, the plug being rotatable between an open position, in which fluid flows through the plug valve, and a closed position, in which fluid flow is obstructed, the cage having first and second opposed flat side surfaces, each side surface including a port;
    - a pair of flow-path inserts respectively having an inner end and an outer end, the inner end of each flow path insert being received in one of the ports, the flow-path inserts defining a flow path through the plug valve that is obstructed by rotation of the plug to the closed position; and
    - a pair of inset seals for sealing the ports when the plug is in the closed position, the inset seals being resiliently biased against the ports in the closed position to provide a fluid-tight seal at the low fluid pressures.
  - 26. The plug valve as claimed in claim 25 wherein the pair of inset seals respectively comprise an elastomeric material bonded to a metal backing and received in respective opposed recesses in the plug, each seal being retained in a respective one of the recesses by a mechanical fastener.
  - 27. The plug valve as claimed in claim 25 wherein the pair of inset seals respectively comprise an elastomeric material received in a recess machined into an outer surface of the plug and adhesively bonded in the recess.