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Butterworth

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(54) **REWINDER APPARATUS AND METHOD**

(75) Inventor: **Tad T. Butterworth**, Ashland, WI (US)

(73) Assignee: **C.G. Bretting Mfg. Co., Inc.**, Ashland, WI (US)

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(58) **Field of Search** 242/521, 532.2, 242/533, 533.7, 541, 541.2, 542, 542.1, 542.2

(56) **References Cited**

U.S. PATENT DOCUMENTS

1,681,046 A	8/1928	Marresford
1,719,830 A	7/1929	Cameron
1,894,253 A	1/1933	McCarthy et al.
1,934,913 A	11/1933	Crisp
2,020,118 A	11/1935	Kellett et al.
2,266,995 A	12/1941	Schultz et al.
2,676,764 A	4/1954	Aulen

(Continued)

FOREIGN PATENT DOCUMENTS

DE	1935584	6/1978
DE	2825154	12/1979
DE	4213712	10/1993
EP	0524158	3/1983
EP	0237903	9/1987
EP	0 867 392	9/1988
EP	0 387 214	9/1990

EP	0 427 408	5/1991
EP	0452284	10/1991
EP	0498039	8/1992
EP	0607525	7/1994
EP	0853060	7/1998
EP	0 872 440	10/1998
FR	2544701	10/1984
GB	1435525	5/1976
GB	2105688	3/1983
GB	2150536	3/1983
IT	1033778	8/1979
IT	1213819	9/1987
IT	1213820	1/1990
IT	1258172	2/1996
IT	1259660	3/1996
WO	WO 94/21545	9/1994
WO	WO 9510472	4/1995
WO	WO 0068129	11/2000

OTHER PUBLICATIONS

“Sincro/Fabio Perini” sales brochure, circa 1994.
PCMC “Magnum” Rewinder, date unknown pamphlet.

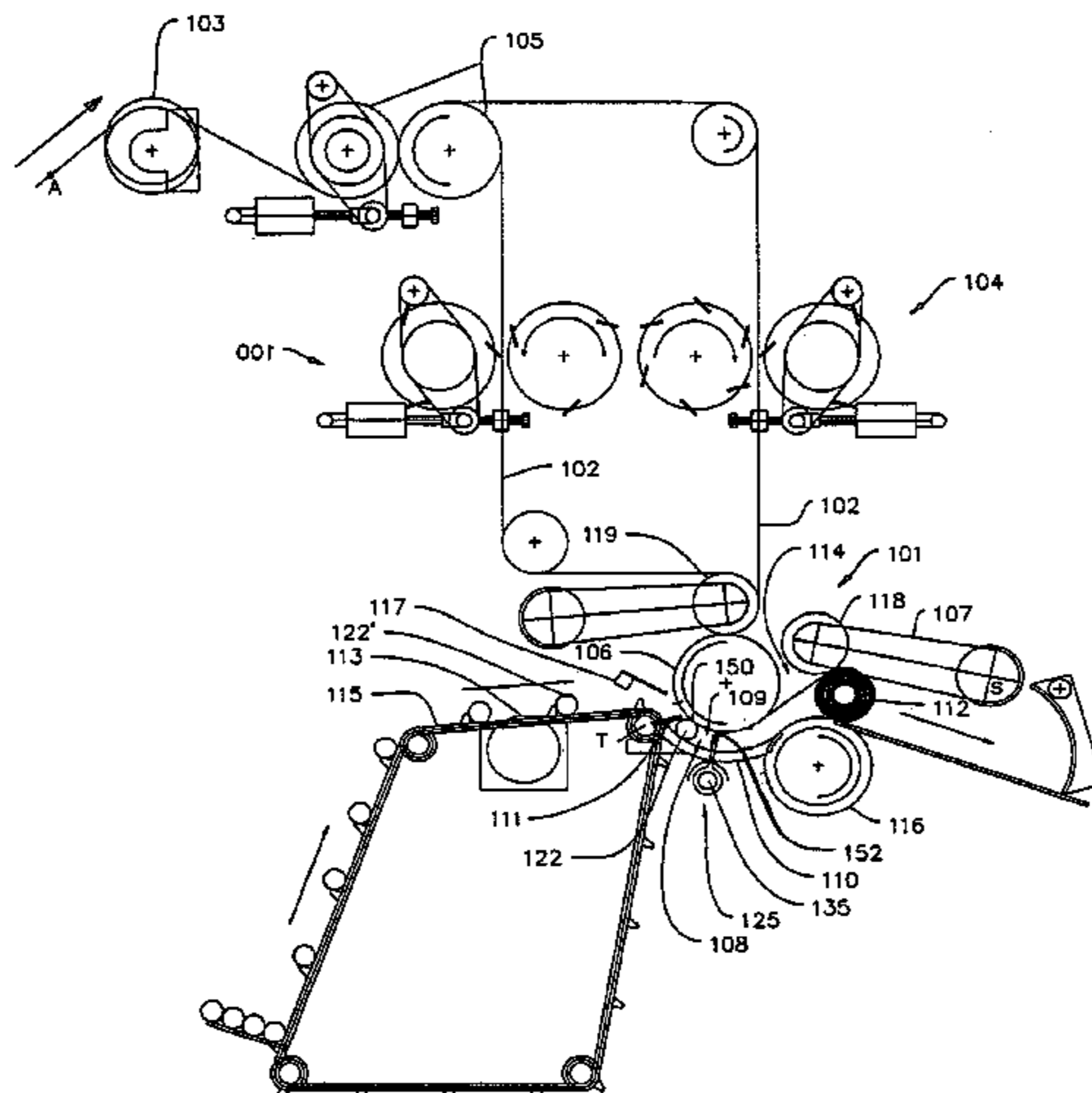
Primary Examiner—William A. Rivera

(74) *Attorney, Agent, or Firm*—Michael Best & Friedrich LLP

(57) **ABSTRACT**

A rewinder having a first winding roll that transports and supports the web, at least one core support plate that is curved for receiving and guiding cores adjacent the first winding roll, and a web separator adjacent the first winding roll and movable into pressing relationship with the web at a velocity at least equal to that of the web. The rewinder of exemplary embodiments winds a web of material a nip defined by the first winding roll, a second winding roll and a rider roll. The web separator of exemplary embodiments comprises a rotatable plurality of fingers that rotates about a common shaft. Because the web separator contacts the web moving at a velocity at least equal to that of the web, the web is effectively separated upstream of the web separator, between the core and the web separator.

64 Claims, 13 Drawing Sheets



U.S. PATENT DOCUMENTS							
2,769,600	A	11/1956	Kwitek et al.	4,496,112	A	1/1985	Olsson et al.
2,775,410	A	12/1956	Schwartz et al.	4,508,279	A	4/1985	Tokuno et al.
2,870,340	A	1/1959	Fransen	4,508,283	A	4/1985	Beisswanger
2,901,191	A	8/1959	Phelps	4,515,321	A	5/1985	Kahlman
2,920,836	A	1/1960	De Bell	4,516,735	A	5/1985	Snygg
3,030,042	A	4/1962	De Gelleke	4,529,141	A	7/1985	McClenathan
3,049,311	A	8/1962	Birch, Jr.	4,541,583	A	9/1985	Forman et al.
3,123,315	A	3/1964	Couzens	4,546,930	A	10/1985	Rohde et al.
3,148,843	A	9/1964	Turner et al.	4,575,018	A	3/1986	Ichikawa
3,179,348	A	4/1965	Nystrand et al.	4,577,789	A	3/1986	Hofmann et al.
3,383,062	A	5/1968	Meihofer et al.	4,583,698	A	4/1986	Nistri et al.
3,389,592	A	6/1968	Bournez et al.	4,588,138	A	5/1986	Spencer
3,471,097	A	10/1969	Phelps	4,601,441	A	7/1986	Oinonen et al.
3,498,558	A	3/1970	Bradley	4,635,867	A	1/1987	Kytonen
3,532,572	A	10/1970	Herman	4,667,890	A	5/1987	Gietman, Jr.
3,549,097	A	12/1970	Seigh	4,687,153	A	8/1987	McNeil
3,552,670	A	1/1971	Herman	4,695,005	A	9/1987	Gietman, Jr.
3,614,010	A	10/1971	Aulen	4,697,755	A	10/1987	Kataoka
3,680,804	A	8/1972	Aaron et al.	4,721,266	A	1/1988	Haapanen et al.
3,697,010	A	10/1972	Nystrand	4,723,724	A	2/1988	Bradley
3,727,853	A	4/1973	Kinoshita	4,775,110	A	10/1988	Welp et al.
3,765,615	A	10/1973	Brink et al.	4,783,015	A	11/1988	Shimizu
3,782,650	A	1/1974	Donnell, Jr.	4,789,109	A	12/1988	Kyytsonen et al.
3,791,602	A	2/1974	Isakson	4,798,350	A	1/1989	Jorgensen et al.
3,791,603	A	2/1974	Lenius	4,807,825	A	2/1989	Eisner et al.
3,794,255	A	2/1974	Harmon et al.	4,828,195	A	5/1989	Hertel et al.
3,817,467	A	6/1974	Dambroth	4,842,209	A	6/1989	Saukkonen
3,823,887	A	7/1974	Gerstein	4,856,725	A	8/1989	Bradley
3,845,914	A	11/1974	Straujups	4,858,844	A	8/1989	Stenquist
3,853,279	A	12/1974	Gerstein	4,874,158	A	10/1989	Retzloff
3,856,226	A	12/1974	Dowd, Jr.	4,875,632	A	10/1989	Kataoka
RE28,353	E	3/1975	Nystrand et al.	4,892,119	A	1/1990	Hugo et al.
3,869,095	A	3/1975	Diltz	4,895,315	A	1/1990	Salmela et al.
3,871,595	A	3/1975	Smolderen	4,909,452	A	3/1990	Hertel
3,881,645	A	5/1975	Kopp	4,919,351	A	4/1990	McNeil
3,889,892	A	6/1975	Melead	4,930,977	A	6/1990	Beeman et al.
3,910,517	A	10/1975	Harrison, III	4,931,130	A	6/1990	Biagiotti
3,926,299	A	12/1975	Bradley et al.	4,932,599	A	6/1990	Doerfel
3,951,890	A	4/1976	Reilly et al.	4,962,897	A	10/1990	Bradley
3,994,396	A	11/1976	Reilly et al.	4,967,804	A	11/1990	Gotti
4,033,521	A	7/1977	Dee	4,977,803	A	12/1990	Blom
4,039,369	A	8/1977	Versteege	4,988,051	A	1/1991	Welschlau et al.
4,055,313	A	10/1977	Yamaguchi et al.	4,997,119	A	3/1991	Meschi
4,123,011	A	10/1978	Kajiwara et al.	5,000,436	A	3/1991	Holmes
4,133,495	A	1/1979	Dowd	5,012,736	A	5/1991	Van Kanegan et al.
4,153,215	A	5/1979	Schulze	5,031,850	A	7/1991	Biagiotti
4,171,780	A	10/1979	Bugnone	5,038,647	A	8/1991	Biagiotti
4,188,257	A	2/1980	Kirkpatrick	5,040,663	A	8/1991	Gould et al.
4,238,082	A	12/1980	Lund	5,040,738	A	8/1991	Biagiotti
4,256,269	A	3/1981	Feighery et al.	5,079,901	A	1/1992	Kotsiopoulos
RE30,598	E	5/1981	Spencer	5,100,040	A	3/1992	Kunreuther et al.
4,265,409	A	5/1981	Cox et al.	5,104,055	A	4/1992	Buxton
4,280,669	A	7/1981	Leanna et al.	5,114,306	A	5/1992	Sjogren et al.
4,284,221	A	8/1981	Nagel et al.	5,137,225	A	8/1992	Biagiotti
4,285,621	A	8/1981	Spencer	5,141,142	A	8/1992	Ramsey
4,327,877	A	5/1982	Perini	5,150,848	A	9/1992	Consani
4,345,722	A	8/1982	Kuhn	5,150,850	A	9/1992	Adams
4,370,193	A	1/1983	Knauthe	5,226,611	A	7/1993	Butterworth et al.
4,408,727	A	10/1983	Dropczynski	5,230,453	A	7/1993	Meschi
4,422,586	A	12/1983	Tetro	5,240,196	A	8/1993	Baarfuesser
4,422,588	A	12/1983	Nowisch	5,248,106	A	9/1993	Biagiotti
4,444,360	A	4/1984	Kaipf et al.	5,249,756	A	10/1993	Biagiotti
4,445,646	A	5/1984	Karr et al.	5,257,898	A	11/1993	Blume
4,448,363	A	5/1984	Mukenschnabl	5,267,703	A	12/1993	Biagiotti
4,460,169	A	7/1984	Bartesaghi	5,271,137	A	12/1993	Schultz
4,485,979	A	12/1984	Dropczynski	5,273,222	A	12/1993	Hutzenlaub et al.
4,485,980	A	12/1984	Gorner	5,273,226	A	12/1993	Dropczynski
4,487,377	A	12/1984	Perini	5,285,977	A	2/1994	Biagiotti
4,487,378	A	12/1984	Kobayashi	5,310,130	A	5/1994	Wieland et al.
4,489,900	A	12/1984	Morizzo	5,312,059	A	5/1994	Membrino
				5,315,907	A	5/1994	Biagiotti

US 6,877,689 B2

Page 3

5,335,871 A	8/1994	Fissmann et al.	5,639,046 A	6/1997	Biagiotti
5,344,091 A	9/1994	Molison	5,653,401 A	8/1997	Biagiotti
5,346,151 A	9/1994	Zimmermann et al.	5,660,349 A	8/1997	Miller et al.
5,357,833 A	10/1994	Biagiotti	5,660,350 A	8/1997	Byrne et al.
5,368,252 A	11/1994	Biagiotti	5,667,162 A	9/1997	McNeil et al.
5,368,253 A	11/1994	Hartley, Jr.	5,690,296 A	11/1997	Biagiotti
5,370,335 A	12/1994	Vigneau	5,690,297 A	11/1997	McNeil et al.
5,377,930 A	1/1995	Noyes	5,722,608 A	3/1998	Yamazaki
5,383,622 A	1/1995	Kohler	5,725,176 A	3/1998	Vigneau
5,387,284 A	2/1995	Moody	5,730,387 A	3/1998	Yamazaki
5,390,875 A	2/1995	Gietman, Jr. et al.	5,732,901 A	3/1998	McNeil et al.
5,402,960 A	4/1995	Oliver et al.	5,746,379 A	5/1998	Shimizu
5,407,509 A	4/1995	Ishizu	5,759,326 A	6/1998	Vigneau
5,409,178 A	4/1995	Stauber	5,769,352 A	6/1998	Biagiotti
5,453,070 A	9/1995	Moody	5,772,149 A	6/1998	Butterworth
5,454,687 A	10/1995	Johnson et al.	5,772,391 A	6/1998	Sjogren et al.
5,460,258 A	10/1995	Tisma	5,785,224 A	7/1998	Nowakowski
5,467,936 A	11/1995	Moody	5,791,248 A	8/1998	Atkins et al.
5,484,499 A	1/1996	Marschke	5,796,221 A	8/1998	Cramer et al.
5,492,287 A	2/1996	Raudaskoski et al.	5,799,467 A	9/1998	Nankervis et al.
5,497,959 A	3/1996	Johnson et al.	5,800,652 A	9/1998	Vigneau et al.
5,505,402 A	4/1996	Vigneau et al.	5,810,279 A	9/1998	Rodriguez et al.
5,505,405 A	4/1996	Vigneau	5,810,282 A	9/1998	McNeil et al.
5,508,279 A	4/1996	Gray	5,820,064 A	10/1998	Butterworth
5,509,336 A	4/1996	Biagiotti	5,839,680 A	11/1998	Biagiotti
5,513,478 A	5/1996	Abt	5,839,688 A	11/1998	Hertel et al.
5,518,200 A	5/1996	Kaji et al.	5,845,867 A	12/1998	Hould et al.
5,518,490 A	5/1996	Ziegelhoffer	5,853,140 A	12/1998	Biagiotti
5,522,292 A	6/1996	Biagiotti	5,901,917 A	5/1999	Schmidt et al.
5,524,677 A	6/1996	Summey, III	5,979,818 A	11/1999	Perini et al.
RE35,304 E	7/1996	Biagiotti	6,000,657 A	12/1999	Butterworth
5,538,199 A	7/1996	Biagiotti	6,050,519 A	4/2000	Biagiotti
5,542,622 A	8/1996	Biagiotti	6,056,229 A	5/2000	Blume et al.
5,544,841 A	8/1996	Didier et al.	6,135,281 A	10/2000	Simhaee
5,558,488 A	9/1996	Gentry	6,328,248 B1	12/2001	Kaipf
5,573,615 A	11/1996	Vigneau et al.	6,354,530 B1	3/2002	Byrne et al.
5,575,135 A	11/1996	Nordstrom	6,422,501 B1	7/2002	Hertel et al.
5,577,684 A	11/1996	Dropczynski et al.	6,479,383 B1	11/2002	Chooi et al.
5,588,644 A	12/1996	Lotto et al.	6,488,226 B2	12/2002	McNeil et al.
5,603,467 A	2/1997	Perini et al.	6,494,398 B1	12/2002	DeMatteis et al.
5,620,151 A	4/1997	Ueyama et al.	6,497,383 B1	12/2002	Daul et al.
5,632,456 A	5/1997	Kruger	6,648,266 B1	11/2003	Biagiotti et al.
5,639,045 A	6/1997	Dorfel			

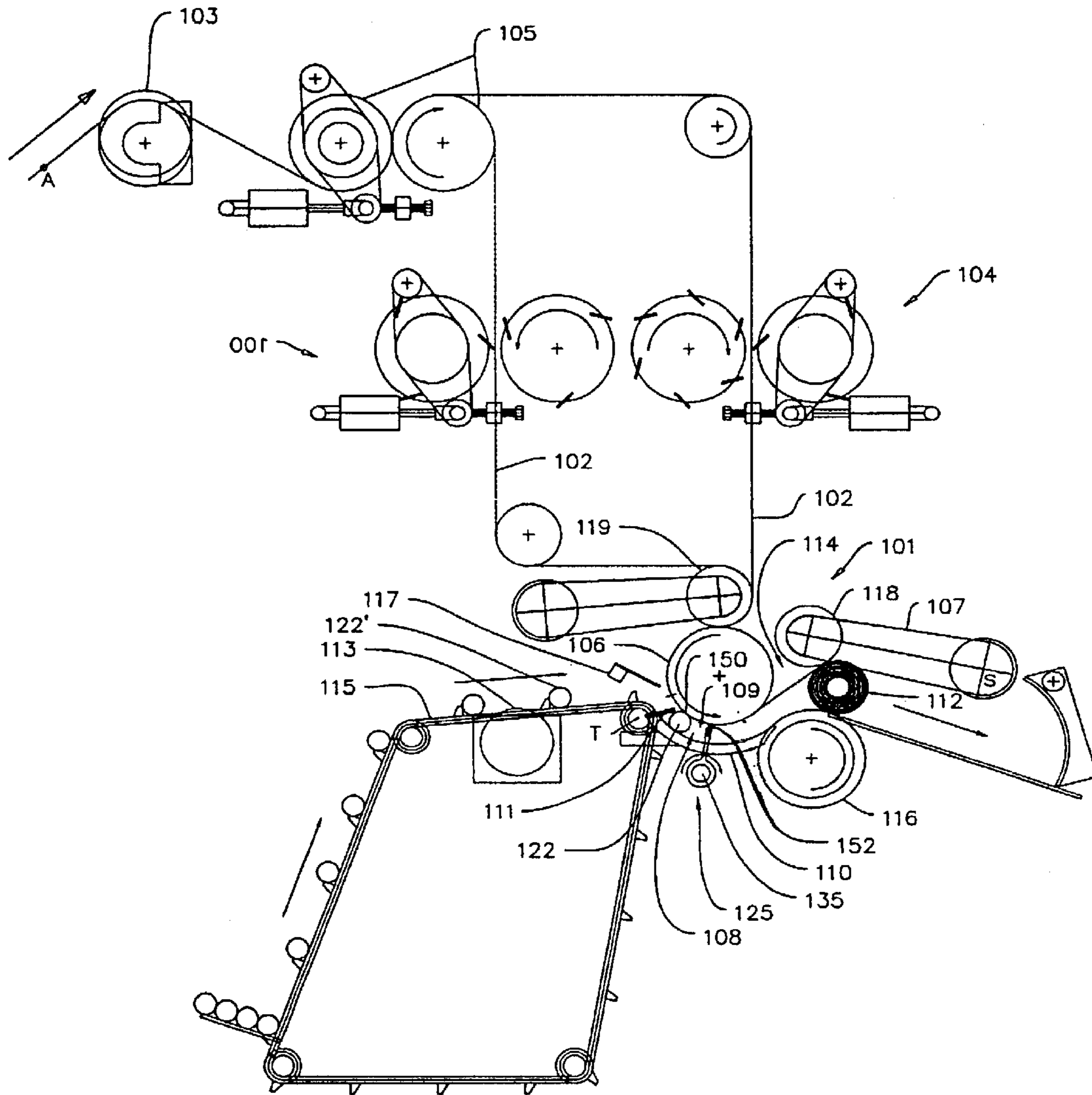


Figure 1

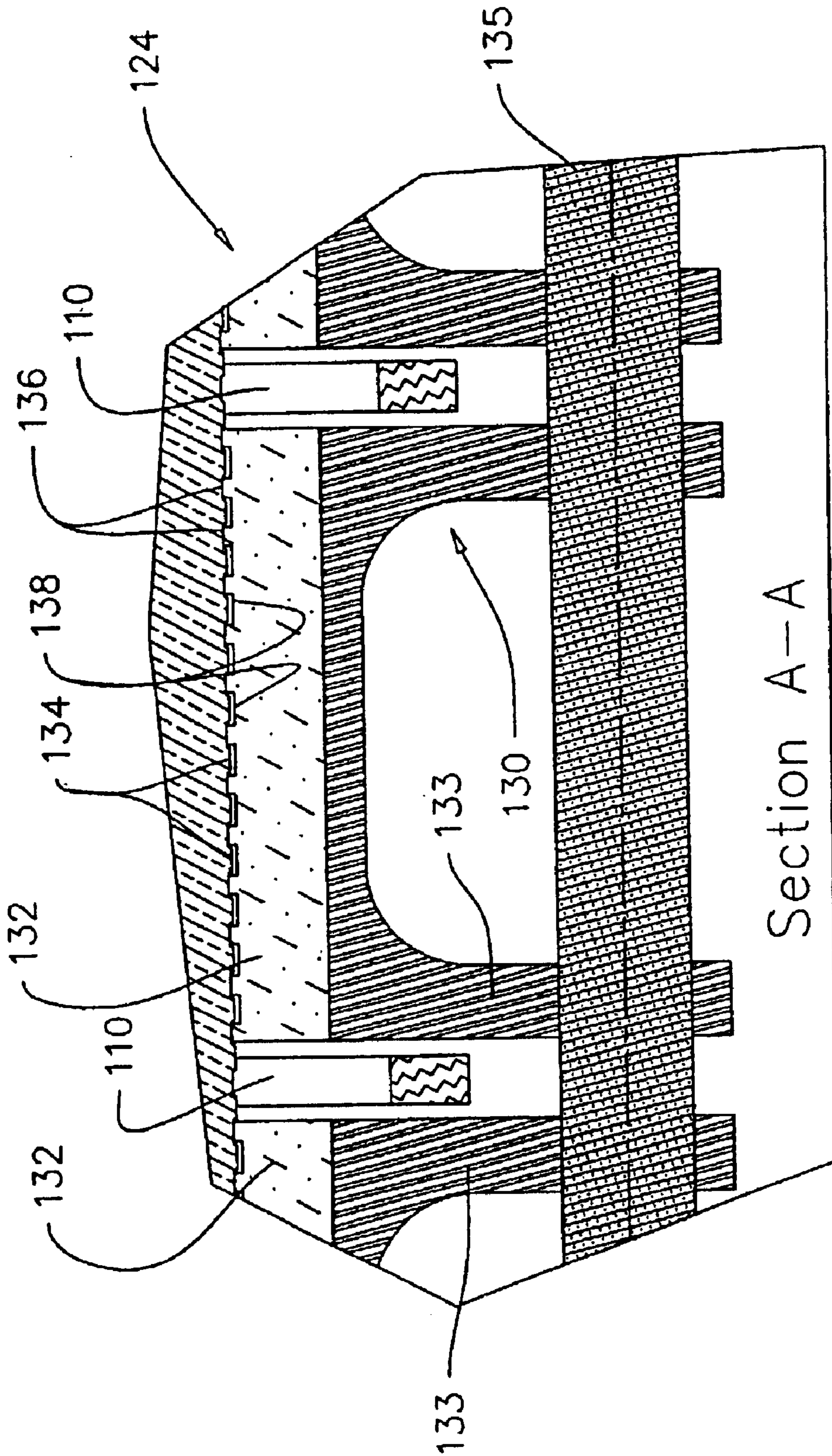
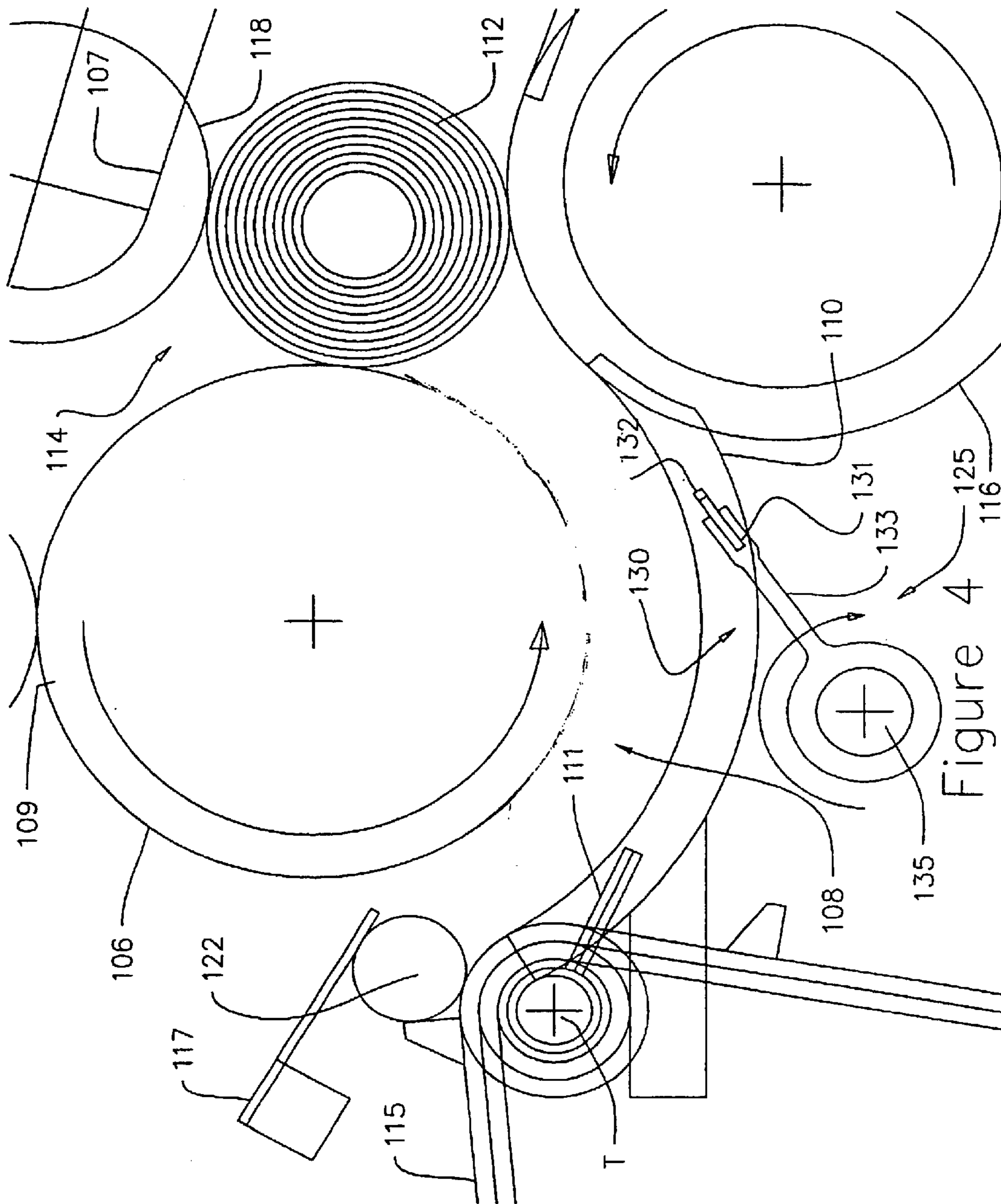
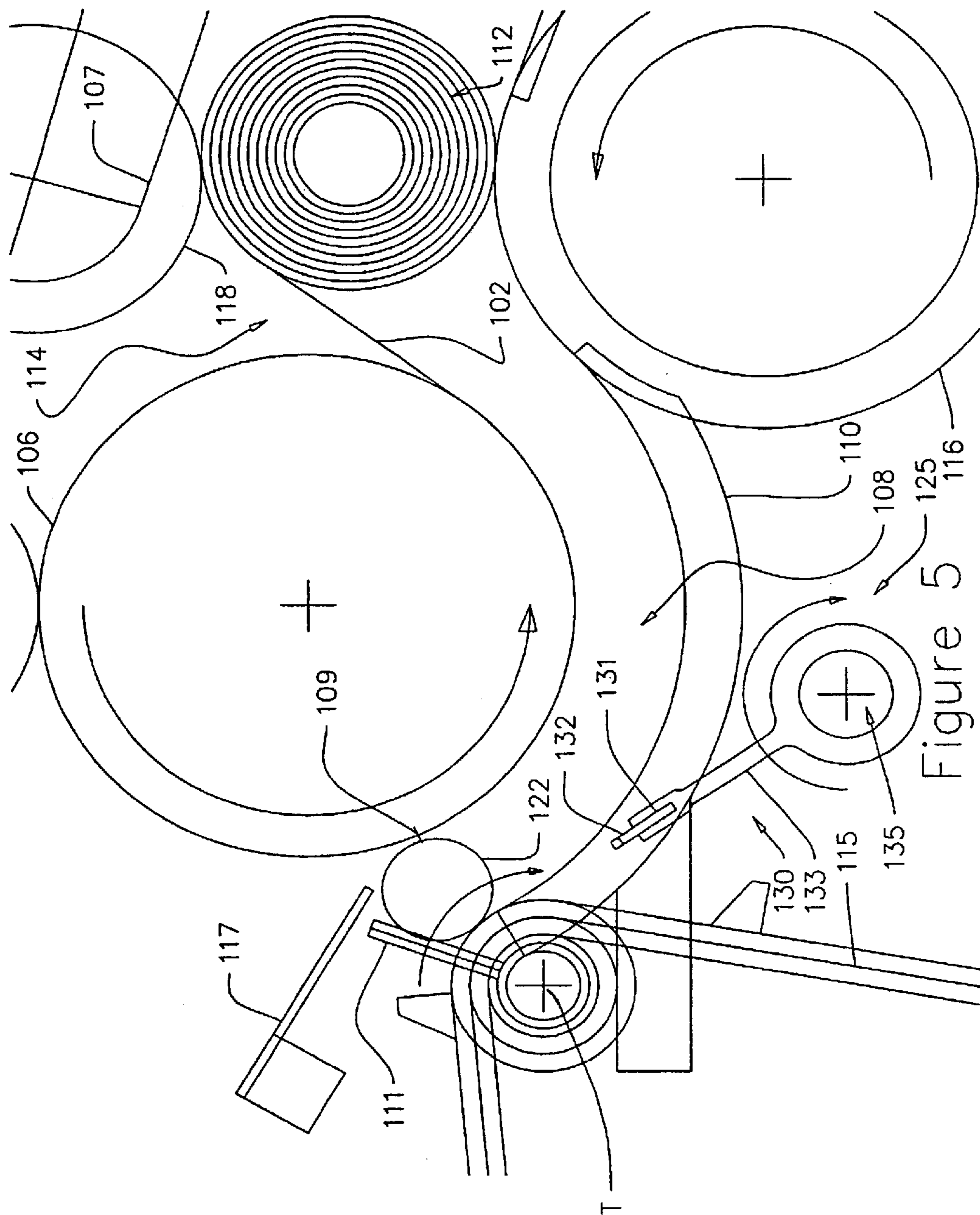
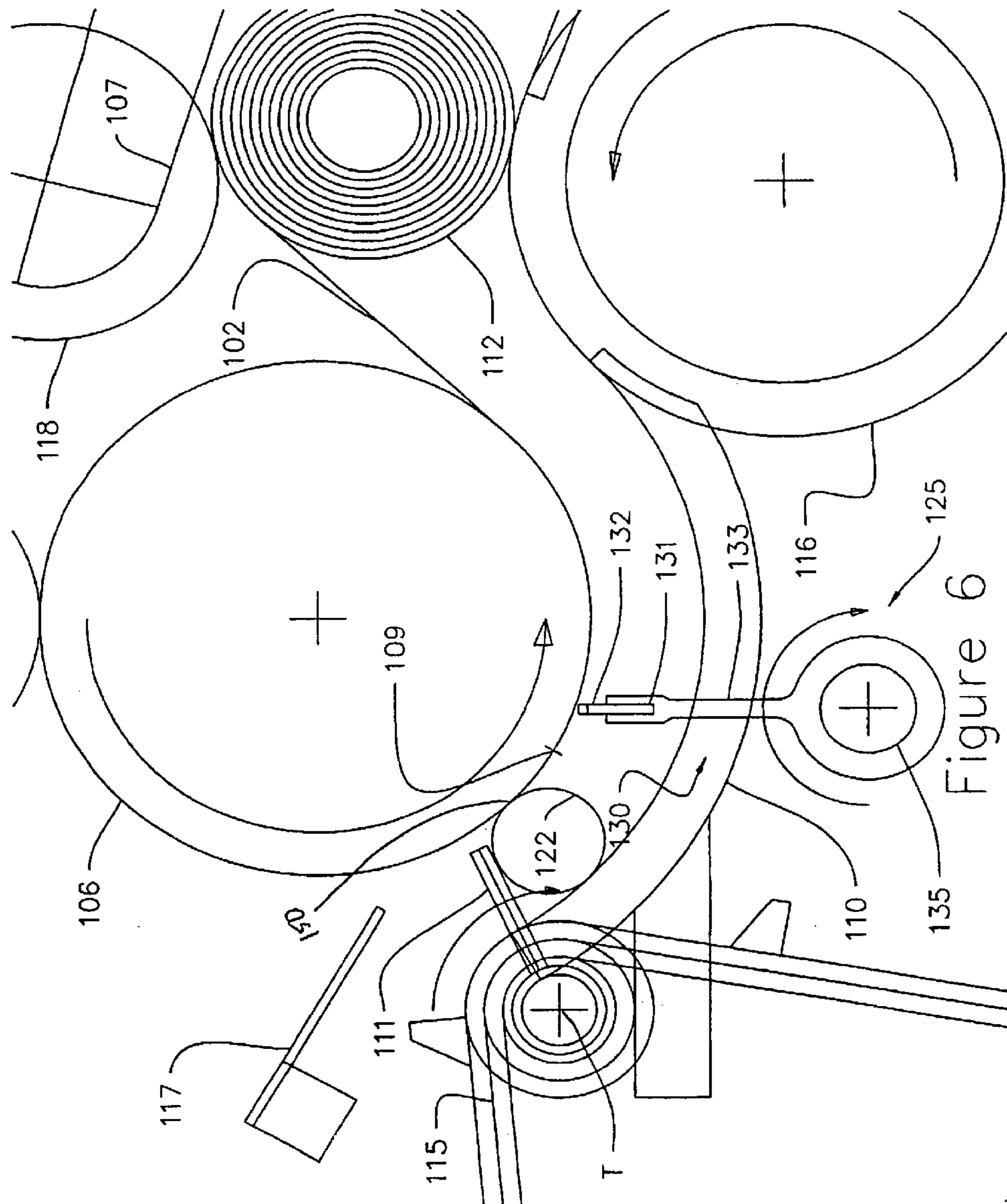
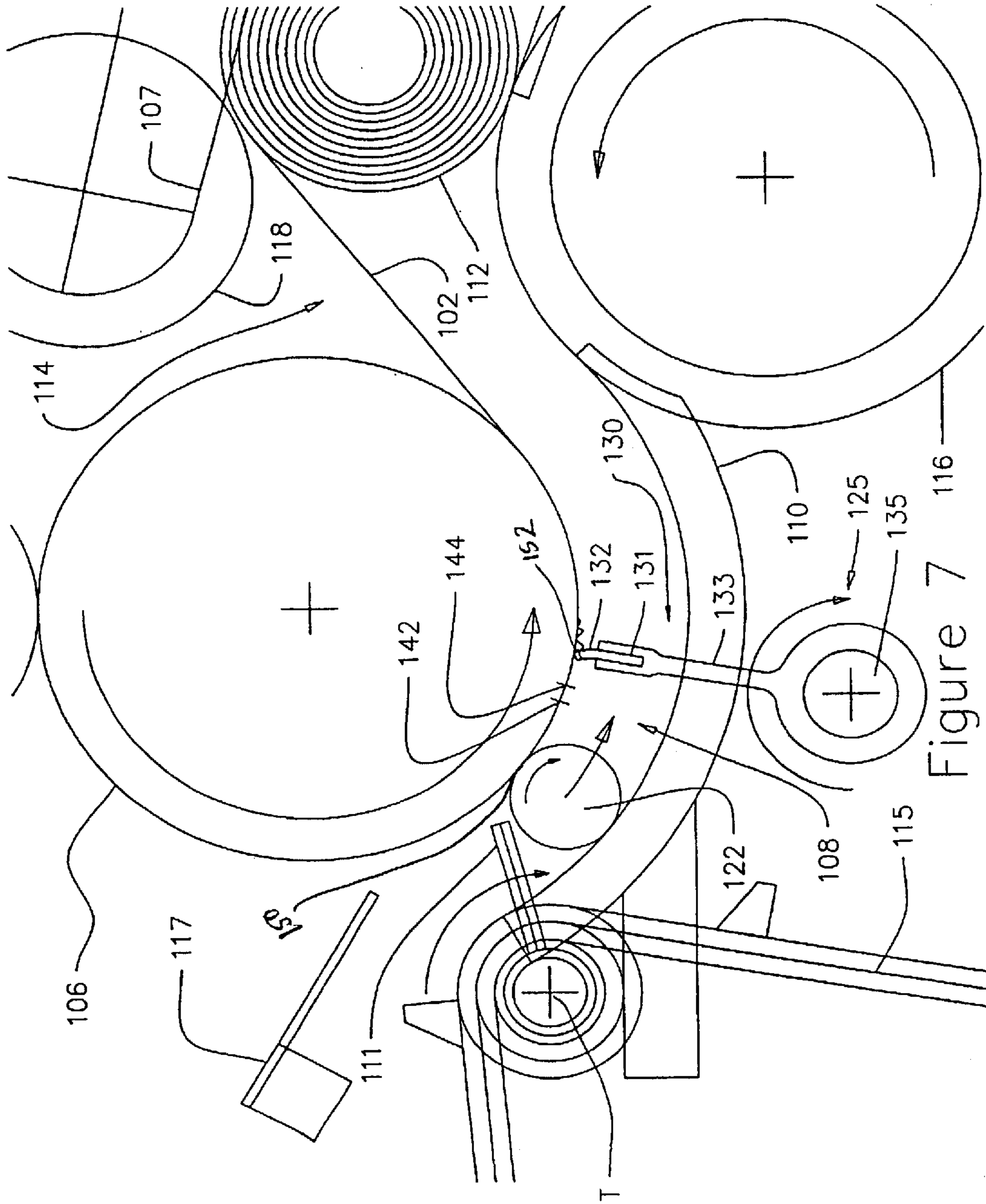


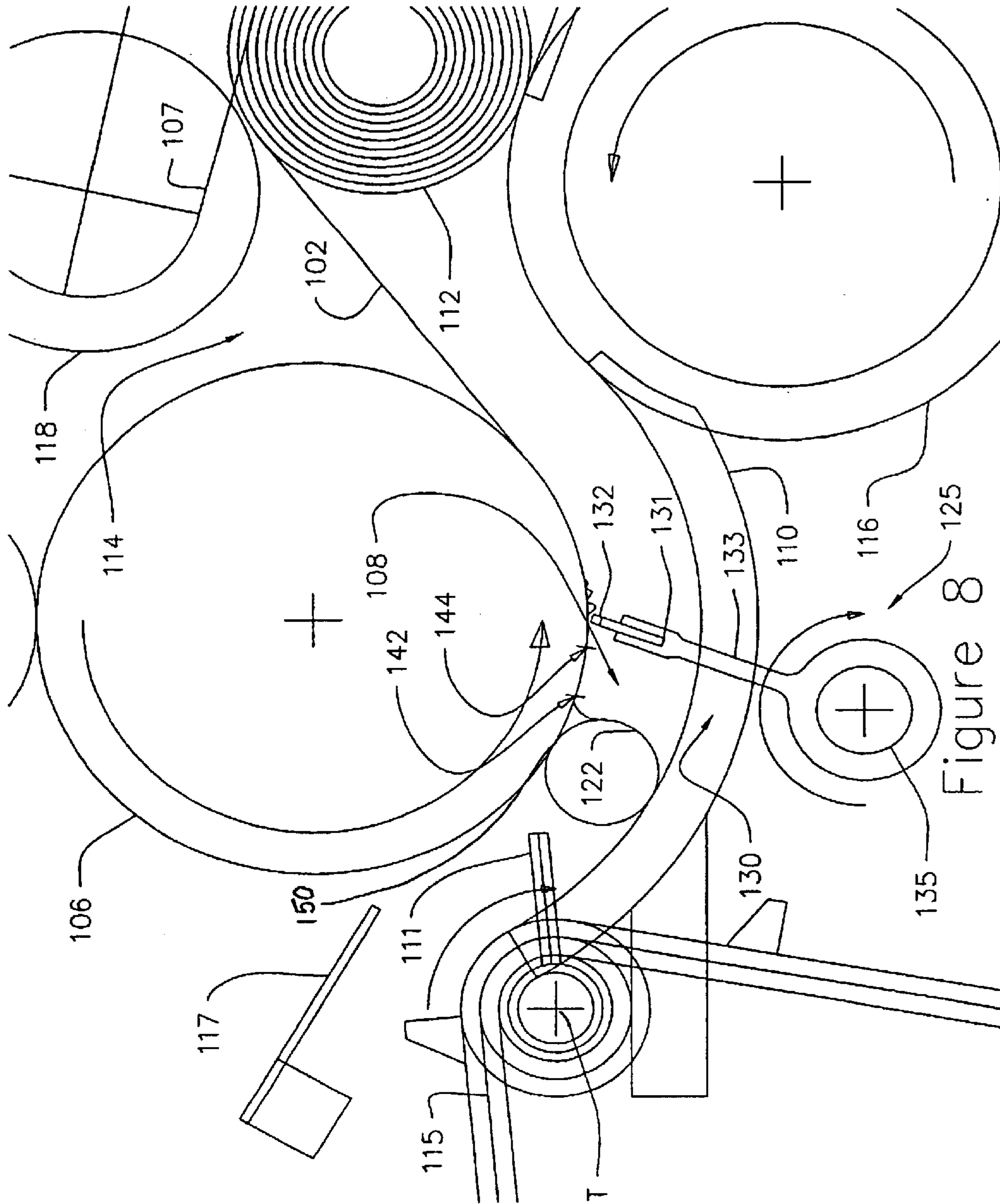
Figure 3











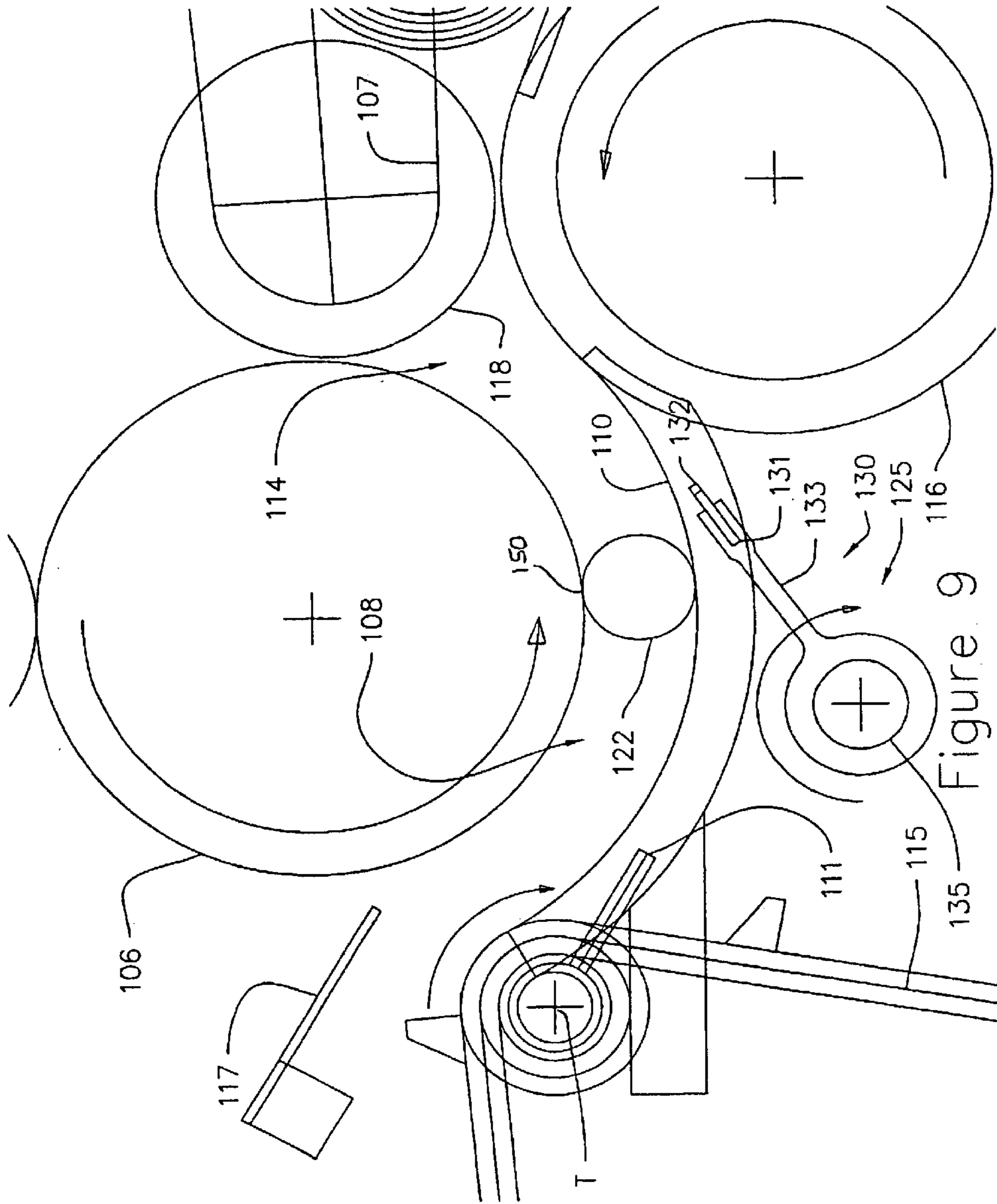
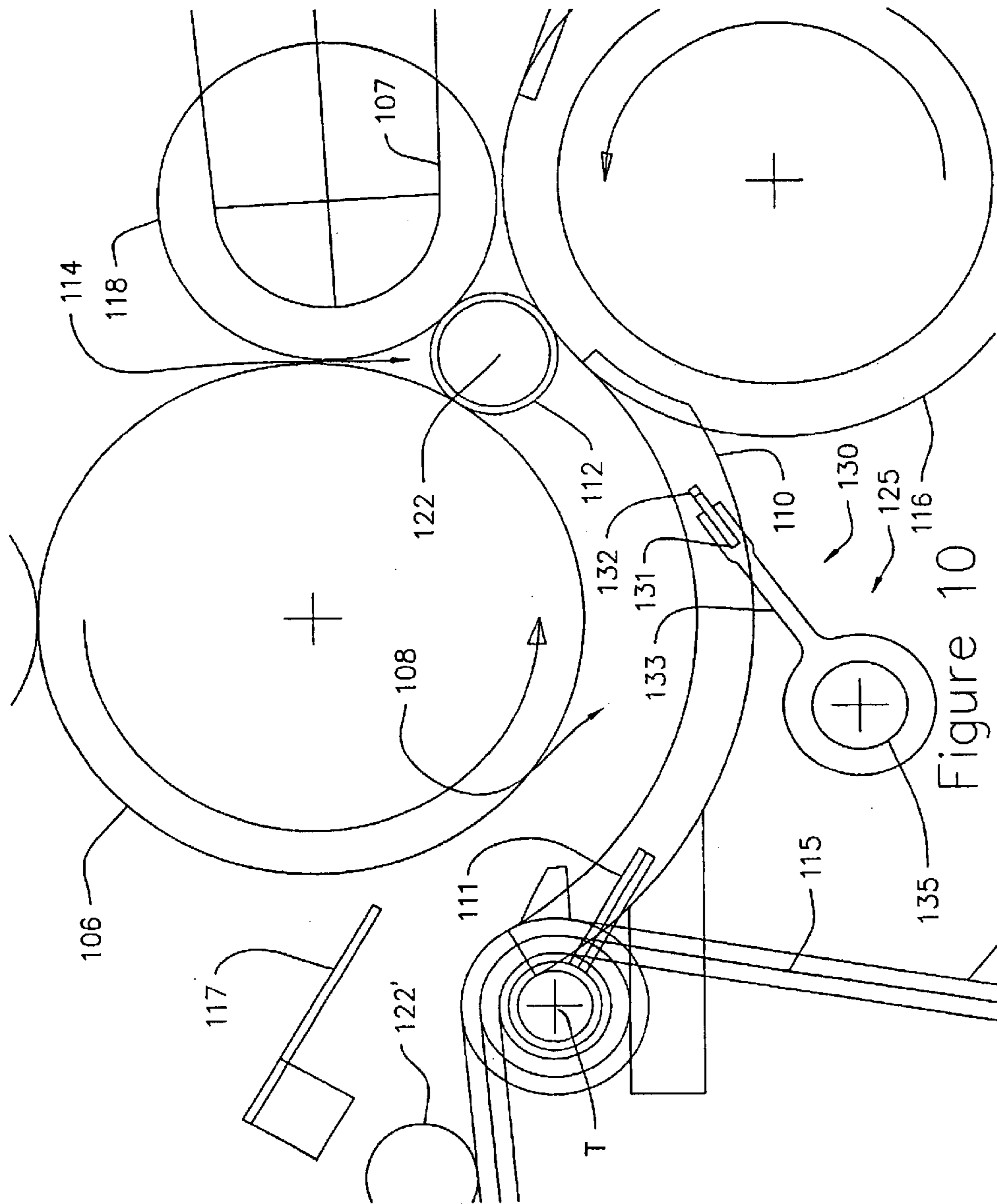


Figure 9



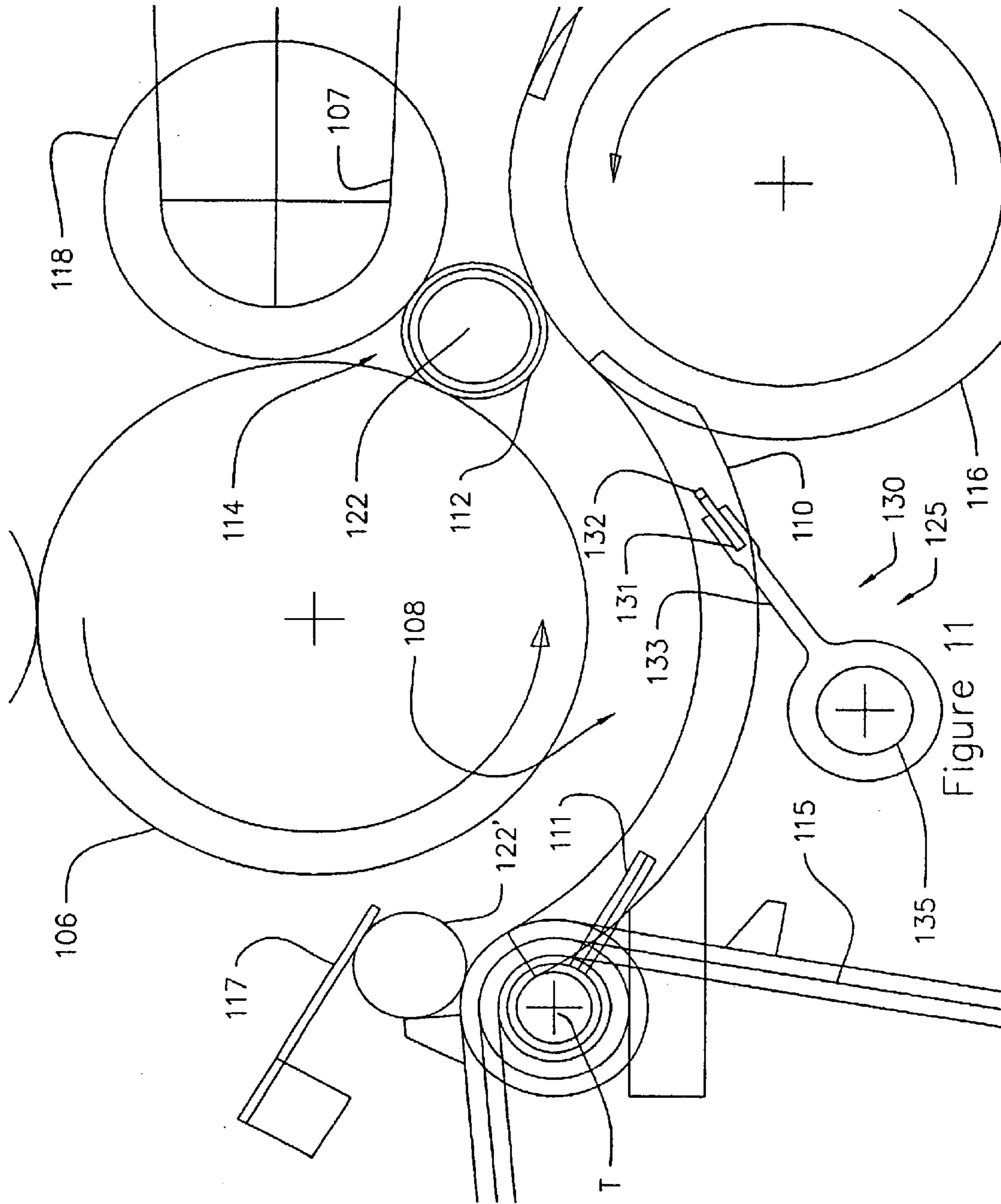


Figure 11

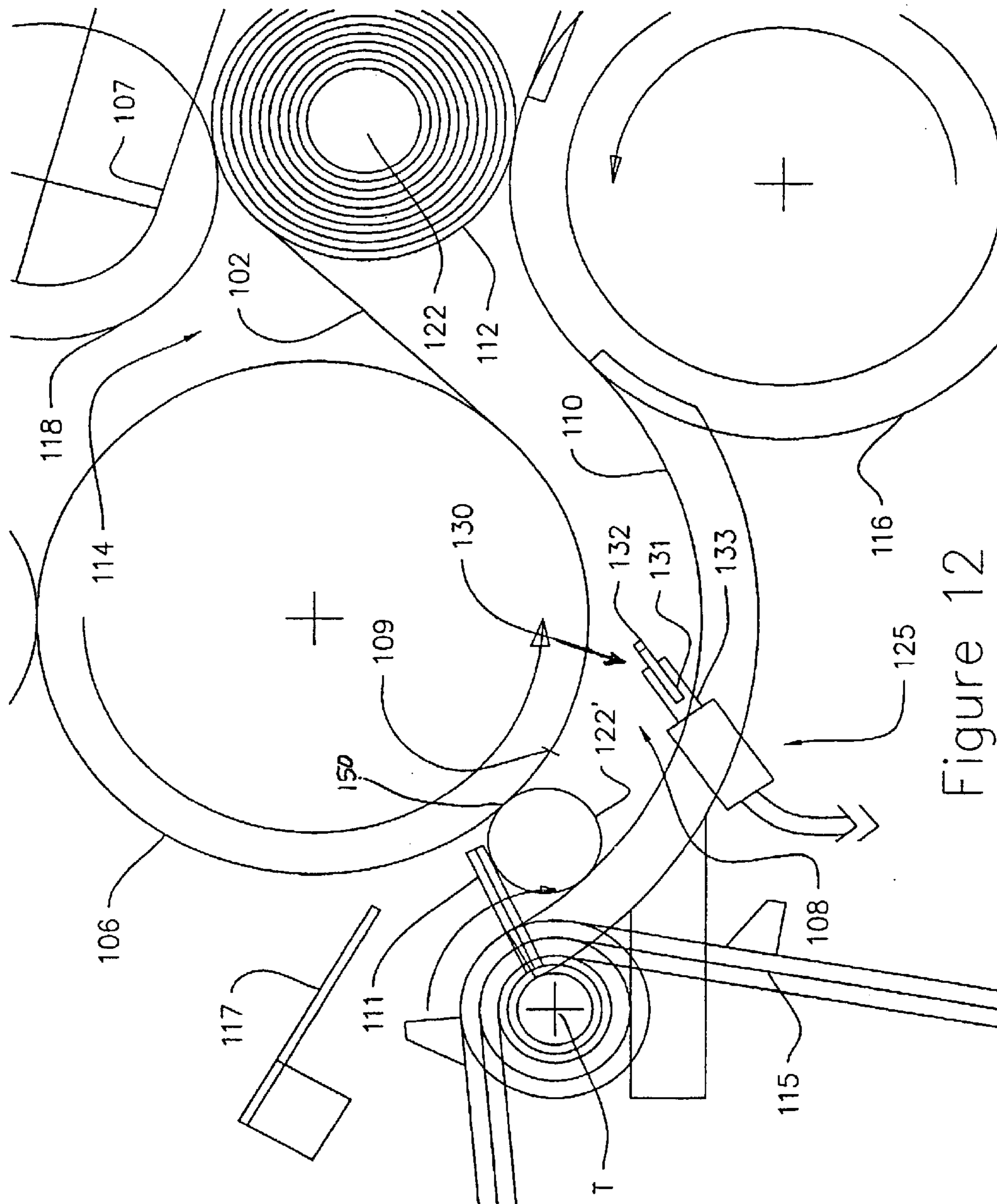


Figure 12

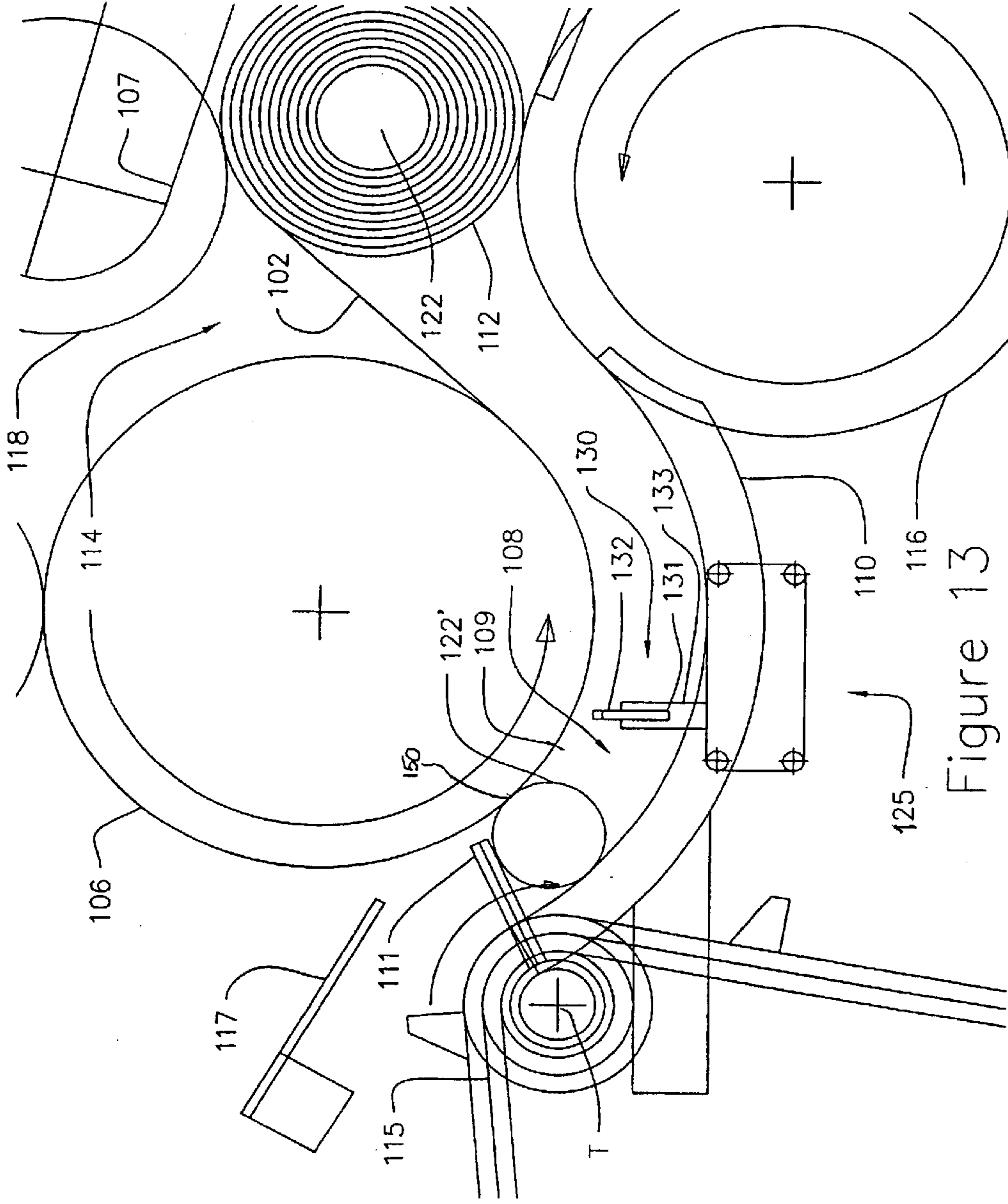


Figure 13

REWINDER APPARATUS AND METHOD**BACKGROUND OF THE INVENTION**

Significant developments in web rewinding have placed ever-increasing product output demands upon web rewinders. Conventional web rewinders are capable of winding a roll or "log" of material in seconds, with maximum winding speeds determined by the strength and other properties of the web and the core upon which the web is wound. Such rewinders are generally limited in their ability to control the position and movement of cores through the rewinder nip, and therefore have limited control over web separation (where cores or core insertion devices perform web separation) and web transfer to new cores. As used herein-after and in the appended claims, the term "nip" refers to an area between two winding elements, such as between two winding rolls, a winding roll and conveyor belt, two facing conveyor belts, or other elements known to those skilled in the art used to rotate and wind a log therebetween. The nip can include an area disposed from the narrowest point between two winding elements, such as when a three-roll winding cradle is employed. The term "web" as used herein and in the appended claims means any material (including without limitation paper, metal, plastic, rubber or synthetic material, fabric, and the like) which can be or is found in sheet form (including without limitation tissue, paper toweling, napkins, foils, wrapping paper, food wrap, woven and non-woven cloth or textiles, and the like). The term "web" does not indicate or imply any particular shape, size, length, width, or thickness of the material. Although faster rewinding speeds are desired, a number of problems arise in conventional rewinders when their maximum speeds are approached, reached, and exceeded. Specifically, the position and orientation of cores entering the winding nip is important to proper web transfer and web separation, but is often variable especially at high rewinder speeds. In some rewinders, a rewinder element separates the web either by pinching the web (thereby creating sufficient web tension between the pinch point and the downstream winding roll to break the web) or by cutting the web. The position and orientation of the core in such rewinders is important to ensuring that the newly-separated web begins to wrap around the core without wrinkling or web damage.

In many conventional rewinders, the web is separated into a trailing edge and a leading edge by a web separating device once the rewound log reaches a predetermined size or sheet count. The trailing edge of the web is wound around the nearly completed log, while the leading edge of the web is wound around a new core that has been positioned near the winding nip. The types of web separating devices vary in form, shape, type of motion and location within the rewinder. In some rewinders, the web is separated by effectively slowing or stopping the motion of the advancing web with the web separating means, thereby causing the web to separate downstream of the web separating means and upstream of the nearly completed log. This type of separation causes the web upstream of the web separating means to develop slack, thus complicating winding of the leading edge of the separated web onto a new core. This type of separation, however, can still be useful depending on the distance between the nearly completed log and the web separating means. If this distance is large relative to the distance between perforations (if a perforated web is employed) reliability and accuracy of web separation can be compromised.

In light of the limitations of the prior art described above, a need exists for an apparatus and method for a web

rewinder in which sufficient core control is maintained to accurately and consistently insert and guide cores toward a rewinder nip, webs can be wound at very high speeds without winding errors, web material can be properly transferred to a newly inserted core, and predictable and reliable web separation is enabled even though significantly different web materials and types are run in the rewinder. Each preferred embodiment of the present invention achieves one or more of these results.

SUMMARY OF THE INVENTION

Some embodiments of the present invention have a first winding surface that transports and supports the web, a core support surface on which cores are guided adjacent the first winding surface toward the web, and a web separator adjacent the first winding surface and movable into and out of pressing relationship with the web at a velocity at least equal to that of the web.

Some embodiments of the present invention wind a web of material adjacent at least one of a first winding roll, a second winding roll and a rider roll, at least two of the first, second, and rider rolls defining a winding nip in the rewinder. The core is moved onto at least one core support surface and guided toward the web running into the winding nip. The web separation bar is moved toward the web at a velocity at least equal to that of a portion of the web adjacent the first winding roll and then contacts and presses the web between the web separation bar and a surface on an opposite side of the web. The web is thus separated into a leading edge and a trailing edge, and the leading edge is wound around a core or mandrel. The core, if employed, can have adhesive applied to it in a number of ways or not at all.

The first winding surface can take a number of different forms, but in some embodiments, takes the form of a winding roll that transports and supports the web. The first winding surface need not transport the web, but if employed, shall provide a surface against which the web can be pressed by the web separator in order to be separated.

The core support surface provides a surface on which cores or mandrels are accurately and consistently guided toward the winding nip, facilitating proper transfer of the leading edge of the separated web onto a new core. Although a variety of different structures can adequately be used for the core support surface in practicing the present invention, some embodiments of the present invention use a plurality of curved plates for supporting and guiding the cores or mandrels adjacent the web and toward the winding nip, in which at least one core support plate is located adjacent the first winding surface.

The web separator moves toward the web at a velocity at least equal to that of the advancing stream of web, and excellent results have been obtained by moving the web separator at a velocity 130% of that of the web. In some embodiments, the web separator comprises one or more fingers, bases and tips. A web separation bar is defined by one or more tips, which contact the web and cause it to separate. The web separator can take a number of different forms, but is shown in the illustrated embodiments to take one of three forms including a rotatable plurality of fingers with tips and bases that rotates about a common shaft; one or more fingers, tips and/or bases mounted onto a linear actuator, specifically a hydraulic or pneumatic cylinder; and one or more fingers, tips and/or bases mounted onto a conveying belt. The web separator, however, need not take any of these forms, but can simply be movable toward and away from a stream of web at a velocity at least equal to that of the web.

Because the web separator employed in the present invention moves toward the web at a velocity at least equal to that of the advancing web, the web is effectively separated upstream of the web separator, between the core and the web separator. Since the distance between the core and the web separator is controlled to be short relative to the distance between perforations in the web (if a perforated web is employed) the present invention allows for accurate, reliable and consistent web separation. Furthermore, the leading edge of the web is not wrinkled and allows for facile and accurate transfer of the leading edge of the web to a new core.

Further objects and advantages of the present invention, together with the organization and manner of operation thereof, will become apparent from the following detailed description of the invention when taken in conjunction with the accompanying drawings, wherein like elements have like numerals throughout the drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

The present invention is further described with reference to the accompanying drawings, which show exemplary embodiments of the present invention. However, it should be noted that the invention as disclosed in the accompanying drawings is illustrated by way of example only. The various elements and combinations of elements described below and illustrated in the drawings can be arranged and organized differently to result in embodiments which are still within the spirit and scope of the present invention.

In the drawings, wherein like reference numerals indicate like parts:

FIG. 1 is an elevational view of the rewinder according to a first preferred embodiment of the present invention;

FIG. 2 is a detail view of the rewinder illustrated in FIG. 1, showing the first and second winding rolls, the rider roll, the core insertion device, the adhesive application area, the core support surface, and the web separator;

FIG. 3 is a cross-sectional view of the rewinder illustrated in FIGS. 1 and 2 taken along line A—A of FIG. 2;

FIGS. 4–11 show a detail view of the winding area of FIG. 2 and the progression of events that occur in the winding area of the rewinder as a core is inserted onto the core support surface and the web is separated and wound around the core;

FIG. 12 shows a detail view of the winding area of FIG. 2 according to a second embodiment of the web separator for the present invention; and

FIG. 13 shows a detail view of the winding area of FIG. 2 according to a third embodiment of the web separator for the present invention.

DETAILED DESCRIPTION OF THE EMBODIMENTS

Referring to the figures, and more particularly to FIGS. 1 and 2, a rewinder constructed in accordance with some of the embodiments of the invention is shown generally at 100. The rewinder 100 includes a number of stations at which various functions are performed. In some of the embodiments, a web 102 of material enters the machine by passing over a bowed roll 103 for minimizing wrinkles in the web 102, then through a set of pull rolls 105 for controlling tension of the web 102. In some embodiments of the present invention, the web 102 then passes through one or more perforation stations 104. Any number of bowed rolls 103, pull rolls 105 or perforation stations 104 can be used without

departing from the present invention. Furthermore, in some embodiments of the invention, no bowed roll 103, pull roll 105 or perforation station 104 is used. For the purpose of example only, one perforation station can be set up for the production of kitchen towels while another station can be set up for bathroom tissue. Other types of perforation stations known to those skilled in the art can be employed without departing from the present invention.

In some embodiments, the web 102 is perforated transversely at one of the perforation stations 104 and is then directed around the ironing roll 119 to a first winding roll 106. Any number of ironing rolls 119 can be used in accordance with the present invention, including an embodiment in which no ironing rolls 119 are used. In the embodiments illustrated in FIGS. 1–13, the web material 102 rewound and separated in this rewinder 100 is periodically perforated, but the web 102 can also be a continuous stream without perforations, or have perforations but not periodic or regular perforations.

As used herein and in the appended claims, the term “upstream” is used to describe any location, element or process that occurs prior to the point or area being referred to; whereas, the term “downstream” is used to describe any location, element or process that occurs ahead of the point or area of reference.

Any upstream equipment or elements for manufacturing, treating, modifying or preparing the web 102 before it reaches the throat 108 can be employed without departing from the present invention. The upstream elements illustrated in FIG. 1 are used only for the purpose of example.

A variety of materials can be rewound satisfactorily using the present invention. As used herein and in the appended claims the term “web” is not limited to tissue, napkin stock, and other paper product, but instead refers to any product found in sheet form, including without limitation, paper, plastic wrap, wax paper, foil, fabric, cloth, textile, and any other sheet material capable of being rewound in the rewinder 100. However, a paper web 102 is described herein for illustrative purposes. The web 102 passes around the first winding roll 106 and into a throat 108 formed between the first winding roll 106 and at least one core support plate 110. As shown in the illustrated embodiment of FIG. 1, a conveyor 115 picks up cores 122 and carries them to an adhesive application area 113. Although shown in the illustrated embodiment of FIG. 1, the use of adhesive is not required in order to practice any embodiment of the present invention. The adhesive, if used, is applied to cores 122 by any of a variety of applicators, including a sprayer, brush, gun, syringe, device for dipping the core into adhesive, and any other similar adhesive applicator or method well-known to those skilled in the art. The conveyor 115 continues moving cores 122 to the winding area 101 of the rewinder 100, as depicted in FIG. 2. A core inserter 111 pushes the core 122 into the throat 108. Other core conveyors, as described below in greater detail, that do not move cores 122 to an adhesive application area or pick up cores 122 but simply deliver cores 122 to the throat 108 can be employed without departing from the present invention. The core conveyor 115 and core inserter 111 described above are presented by way of example only.

In some embodiments of the present invention, paper logs 112 are wound in a nip 114 between the first winding roll 106, a second winding roll 116 and a rider roll 118 as known in the art, although the invention also offers advantages in other rewinding processes, including winding the web 102 partially or fully around a core 122 in the throat 108,

winding the web 102 between two side-by-side rolls without the use of a rider roll, and any other orientation or combination of winding rolls or core support plates 110 capable of winding the web 102 around a core 122 or mandrel. If employed, the rider roll 118 is movable from a position close to the winding rolls 106, 116 when the log 112 is relatively small to a position away from the winding rolls 106, 116 as the diameter of the log 112 increases. Many different devices can be used to move the rider roll 118, including a pivot arm 107 pivotable about a first axis S, an accordion-style system of bellows that is compressed as the diameter of the log 112 increases, a fixed or movable cam member with an aperture or surface upon which an extension of the rider roll 118 follows as the diameter of the log 112 increases, and any other equipment or element capable of moving the rider roll 118 away from the other rolls 106, 116 to accommodate an increasing log 112 diameter. The pivot arm 107 and first axis S are shown in FIG. 2 only for exemplary purposes.

While roll structures are illustrated in FIGS. 1, 2 and 4-13 and described herein, belts and other mechanisms, as described in greater detail below, capable of transporting the web 102 to the throat and winding the web 102 can also be used satisfactorily without departing from the present invention. For example, the web 102 can be wound around a moving belt, moving in a circular path or otherwise, instead of the first winding roll 106.

Referring to FIGS. 1-13, at least one core support plate 110 receives and guides cores 122 into and through the throat 108 toward the nip 114, while a web separator 125 generates separation of the web 102. The web separator 125 has one or more fingers 130, bases 133 and tips 132. A web separation bar 124 (see FIG. 3) is defined by one or more tips 132 (or bases 133 if no tips 132 are used, or if the tips are integrally part of the bases). While the embodiments illustrated in FIGS. 1-13 use cores 122, it will be apparent that the present invention is useful for winding coreless products using mandrels or other winding initiation devices as well. Accordingly, the disclosure herein referring to the use of the cores 122 in rewinding operations of the present invention is equally applicable to the use of mandrels in such operations.

The web separator 125 can take a number of different forms, as described below in greater detail. In the illustrated embodiment of FIG. 3, the web separator 125 is composed of a plurality of elongated web separation fingers 130 arranged on and extending radially from a common shaft 135 that runs transversely in the rewinder 100, but the web separator 125 can be located on any number of different shafts or other rotatable elements as desired. The fingers 130 allow for the passage of at least one core support plate 110 therebetween by providing a plurality of open spaces between each finger 130 through which at least one core support plate 110 can move. Additionally, in some embodiments, the web separation bar 124 is movable into and out of the throat 108 to contact the web 102 adjacent the first winding roll 106 at a velocity at least equal to that of a portion of the web 102 adjacent the first winding roll 106. In some embodiments, the web separation bar 124 is mounted for rotation into and out of the throat 108. Additionally, in some embodiments, the motion of the web separation bar 124 is generally directed clockwise with reference to FIG. 2, but can also be directed counterclockwise with reference to FIG. 2, or intermittently clockwise, then counterclockwise, or vice versa, or can be rotated, pivoted, or moved in any other manner to bring the web separation bar 124 into contact with the web 102 at a greater velocity than the portion of the web 102 adjacent the first winding roll 106.

As shown in FIG. 6, at least one resilient tip 132 of the web separation bar 124 on a base 133 (or base 133 if no tip

is used, or if the tip is integrally part of the base) pinches the web 102 between the resilient tip 132 and the first winding roll 106 downstream of the core 122. The one or more tips 132 can comprise a variety of resilient or rigid materials. In some embodiments, the tip 132 comprises polyurethane having a durometer of between sixty and one hundred, although other materials, such as polyurethane having a durometer outside of the aforementioned range, rubber, silicone, ultra-high molecular weight poly(ethylene), aluminum, steel, and any other material capable of contacting and separating the web 102 can also be employed without departing from the present invention. Furthermore, the tip can be mounted to a base 133 of the web separator 125 in any manner. The tip 132 can be mounted directly to the base 133 as illustrated in FIGS. 1, 2 and 4-11 by placing the tip 132 inside a recess 131 in the base 133 and bolting the tip 132 in place, or by having one side of the base 133 removable and bolted back in place over the tip 132 capturing the tip in the recess 131, or by clamping the base 133 closed over the tip 132 either with additional clamps or by having the base 133 itself function as the clamp, or by fitting the tip 132 into the recess 131 of the base 133 with a snap-fit between at least one rib on the tip and at least one groove in the recess 131 of the base 133, or by defining the entire web separation finger 130 with the tip 132, provided a sufficiently durable material is used.

Alternatively, the tip 132 can be spring mounted to the base 133 to provide resilience. For example, a variety of materials can be coupled between the tip 132 and the base 133, including without limitation one or more compression springs, one or more blocks and/or layers of rubber, polyurethane, silicone, and any other material capable of providing resilience to the tip 132. The resilient nature of the tip 132 in some embodiments enables tolerances for the interference between the first winding roll 106 and the tip 132 to be less restrictive while maintaining product quality and performance.

In some embodiments, the one or more resilient tips 132 of the web separation bar 124 travel through a circular path, represented by a dash-dot circle in FIG. 2, intersecting or tangent to the path traveled by the advancing stream of web. In some embodiments, the web separation fingers 130 are arranged on a common shaft 135 running transversely in the rewinder 100, but can be located on any number of different shafts or other rotatable elements as desired. In other embodiments, the one or more resilient tips 132 of the web separation bar 124 travel through a non-circular path, such as a path that is substantially triangular, rectangular, square, straight, arcuate, and the like. It will be apparent to one of ordinary skill in the art that any path shape can be used, provided the one or more resilient tips 132 contact the web at the desired location.

FIG. 3 shows that the first winding roll 106 of the illustrated embodiments comprises alternating annular rings of a high friction surface 134 and a low friction surface 136 spaced transversely; that is, some rings have a higher coefficient of friction than others. The annular rings of the first winding roll 106 can be arranged in any pattern, but the rings are shown as alternating rings of high friction surface 134 and a low friction surface 136 for the purpose of example only. However, any ratio of high to low friction surface areas across the roll can be used. The high friction surfaces 134 are shown as ridges for clarity in the exemplary embodiment illustrated in FIG. 3, although in some embodiments the high friction surfaces 134 would be raised only slightly above that of the low friction surfaces 136. One or more of a number of different materials can be used for the

high friction surfaces **134**, including without limitation emery cloth; rubber; polyurethane; any knurled or embossed surface; unpolished wood, natural or otherwise, and any other material with a higher coefficient of friction than the material used on other rings of the first winding roll **106**. Similarly, one or more of a number of different materials can be used for low friction surfaces, including without limitation poly tetrafluoroethylene (PTFE); ultra-high molecular weight polyethylene; polished steel; aluminum; silicone; polished wood, natural or otherwise; and any material with a lower coefficient of friction than the accompanying higher friction surface material chosen. Thus, any combination of materials can be used for the annular rings on the first winding roll **106** where the materials chosen for some of the rings have a higher coefficient of friction than the materials chosen for the other rings.

In some embodiments of the present invention, the one or more resilient tips **132** of the web separation bar **124** comprise recessed areas **138** to prevent contact of the one or more resilient tips **132** with the high friction surfaces **134** of the first winding roll **106**. Although FIG. **3** illustrates an embodiment where the tips **132** have recesses **138** to accommodate the high friction surfaces **134**, tips **132** with no recesses **138** or tips **132** with recesses **138** that do not accommodate the high friction surfaces **134** of the first winding roll **106** are also well within the spirit and scope of the present invention. Upon intersecting the path of the web **102**, which is advancing adjacent the first winding roll **106**, as shown in FIGS. **1–13**, the web separation bar **124** contacts the web **102** and pinches it against the first winding roll **106** adjacent only the low friction surfaces **136**, when low friction surfaces **136** are employed. The web separation bar **124** accelerates to a velocity at least equal to that of the web adjacent the web separation bar **124** at the time of separation. In some embodiments, the web separation bar **124** is accelerated through rotation. The web separation bar **124** can be accelerated through any angle sufficient to generate any velocity at least equal to that of the velocity of the web **102** adjacent the web separation bar **124** at the time of separation. In some embodiments, the web separation bar **124** can be accelerated through 270° of rotation; however other angles through which the web separation bar **124** is accelerated are possible and fall within the spirit and scope of the present invention. By way of example only, the web separation bar **124** can be accelerated to a velocity at least 100% of that of the web adjacent the web separation bar **124**. In other embodiments, the web separation bar **124** can be accelerated to a velocity at least 125% of that of the web adjacent the web separation bar **124**. In still other embodiments, the web separation bar **124** can be accelerated to a velocity at least 150% of that of the web adjacent the web separation bar **124**. However, excellent results can often be achieved by accelerating the web separation bar **124** to a velocity at least 130% of the web adjacent the web separation bar **124**. Still, other web separation bar velocities can be used, each falling within the spirit and scope of the present invention.

In some embodiments of the present invention, the web separation bar **124** is timed to contact the web **102** at a position between perforations **109**, when a perforated web **102** is used. At the point of contact with the web separation bar **124**, the web **102** adjacent the web separation bar **124** is rapidly accelerated to the web separation bar speed and slips on the first winding roll **106** due to the high coefficient of friction between the web separation bar **124** and the web **102**. The velocity of the web **102** adjacent the first winding roll **106** and the velocity of a point on the surface of the web

separation bar **124** can be the same or substantially the same for a fraction of a second to perform the functions of separating the web as described in greater detail below. However, this amount of time can be longer depending upon the speed of the first winding roll **106**, the web **102**, and the web separation bar **124** (i.e., with slower speeds of these elements). The amount of time these velocities are the same will typically depend at least partially upon the interference between the web separation bar **124** and the roll **106** and the respective velocities of the bar **124** and the roll **106**. The contact point or line between the web separation bar **124** and the web **102** adjacent the first winding roll **106** can be referred to as a web control point **152** in which the velocity of the web is positively controlled and known. In FIGS. **1, 2** and **4–13**, the web control point **152** is shown as a region within which the web control point **152** will be located. Tension in the web **102** between the web separation bar **124** and the core **122** increases above the tensile strength of a perforation **109** in the web **102**. Because the web separation bar **124** is close to the core **122** when the web separation bar **124** contacts the web **102**, only one perforation **109** exists between the web separation bar **124** and the core **122** in some embodiments. In other embodiments, more than one perforation **109** can exist in the area between the web separation bar **124** and the core **122**. Locating at least one perforation **109** in this area of high tension helps ensure that the web **102** will separate on the at least one desired perforation **109**, unlike some winders that include a web separator **125** operating at a speed slower than that of a portion of the web adjacent the first winding roll **106**. This controlled separation of the web **102** helps guarantee that each log **112** has a desired number of sheets or has a more accurate sheet count, substantially reducing costs of surplus sheets commonly resulting from operation of prior art devices.

In some embodiments of the present invention, the core support plate **110** comprises aluminum. Other materials can be employed for the core support plate, including without limitation steel, ultra-high molecular weight poly(ethylene), or any other material capable of supporting a core **122** or mandrel as it approaches the web **102**. One or more core support plates **110** can be used in the present invention. Multiple core support plates **110** are used in the illustrated embodiments, as shown in FIG. **3**, but only one is shown in FIGS. **1, 2** and **4–13**. In the illustrated embodiments, the rewinder **100** has multiple core support plates **110** that are curved, the set of which extends in at least part of the rewinder **100**. The multiple core support plates **110** are spaced apart sufficiently to permit one or more web separation fingers **130** to pass between adjacent plates **110** (FIG. **3**). In some embodiments, the curve of the core support plate **110** follows a portion of the first winding roll **106** concentrically and in some cases extends from the location where cores **122** are inserted into the winding area **101** to the second winding roll **116**. In some embodiments, as the core **122** is inserted onto the core support plate **110**, the core **122** is driven by the first winding roll **106** while rolling along the core support plate **110** toward the winding nip **114**. In other embodiments, the core **122** rolls freely along the core support plate **110**. In still other embodiments, the core support plate **110** takes a different form altogether and the core **122** is brought to the vicinity of the web **102** by different devices, as discussed below in greater detail. Thus, when the core **122** rolls along the core support plate **110** while being driven by the first winding roll **106**, the average velocity of the core **122** along the core support plate **110** is approximately 50% of the velocity of the web **102** adjacent

the first winding roll 106. However, when other forms of a core support plate 110 are employed, the core 122 can move toward or adjacent the web 102 at other velocities or can approach the web 102 by other devices.

In some embodiments, as shown in FIGS. 1, 2 and 6–13, the distance between the core support plate 110 and the surface of the first winding roll 106 is less than the diameter of the cores 122, helping to provide proper alignment of the core 122 as it proceeds along the core support plate 110 toward the winding nip 114 and causing the core 122 to deflect slightly, in turn, providing pressure between the core 122 and the web 102 adjacent the first winding roll 106. With continued reference to the illustrated embodiments, this pinching action between the core 122 and the web 102 forces the web 102 against the high friction surfaces 134 of the first winding roll 106. Forcing the web 102 against the high friction surfaces 134 helps assure that the velocity of the web 102 at the point of contact with the core 122 will be the same or substantially the same as the velocity of a point on the surface of the first winding roll 106 adjacent the web 102. The contact point or line between the core 122 and the web 102 adjacent the first winding roll 106 can be defined as a web control point 150 in which the velocity of the web 102 is positively controlled and known. In FIGS. 1, 2 and 4–13, the web control point 150 is shown as a region within which the web control point 150 can be located.

However, in some embodiments of the present invention, the core 122 does not press against the first winding roll 106 (with the web 102 therebetween) with sufficient force to define the web control point 150. In other words, the web 102 is not necessarily sufficiently retained at the location of the core 122 to define a location where the speed of the web 102 is the same or substantially the same as that of the first winding roll 106. Accordingly, in some embodiments and/or for a period of time or movement of the core, there need not necessarily be a web control point 150 at the core 122. In these embodiments, it is not necessary for the core 122 to press against the web 102 with the force described above, because the amount of web wrap around the curved surface of the first winding roll 106 generates sufficient tension in the web 102 to separate the web 102 along a row of perforations 109 lying upstream of the point or line of contact between the web separation bar 124 and the web 102. Furthermore, by employing embodiments in which a web control point 150 is not necessary, lighter cores 122 can be used in the rewinder 100, and/or the cores 122 used in the rewinder 100 do not need to be compressed as much or be able to withstand as great of force while proceeding toward the winding nip 114.

In some embodiments of the present invention, there are two web control points 150, 152 in this rewinding process: one web control point 150 being the contact between the core 122 and the web 102 adjacent the first winding roll 106, and another web control point 152 being the contact between the web separation bar 124 and the web 102 adjacent the first winding roll 106. The web is stretched in the area between the two control points 150, 152. The amount of stretch is determined by the relative velocity difference between the two web control points 150, 152 and the duration of contact at the web separation bar web control point 152. The combination of velocity difference and contact duration is enough to rupture the perforation 109 located in this high-tension zone between the web control points 150, 152.

In some webs 102 employed in the present invention, web stretch and perforation bond strength can be highly variable. In some embodiments of the present invention, different operating conditions can be allowed by making both the

relative velocity and the contact duration adjustable, helping the rewinder 100 accommodate a wide range of web materials. The web separation bar 124, the conveyor 115 and the core inserter 111 can be driven by one or more of a number of driving devices or actuators, including without limitation programmable electric, hydraulic, or pneumatic motors, solenoids, linear actuators, and the like, driven directly or indirectly via belts and pulleys, chains and sprockets, one or more gears, and any other driving device or actuator capable of facilitating the timing of the web separation bar 124, the conveyor 115 and the core inserter 111 and helping to ensure the presence of the desired number of perforations 109 in the zone between the two web control points 150, 152.

FIGS. 4–11 are detailed views of the exemplary rewinder 100 illustrated in FIGS. 1–3, showing the progression of events in the winding area 101. FIG. 4 shows a log 112 being wound in the winding nip 114 between the first winding roll 106, the second winding roll 116, and the rider roll 118. A core 122 is positioned on the chain conveyor 115 near the entrance to the throat 108, between the first winding roll 106 and the core support plates 110. The conveyor 115 and core inserter 111 can be timed and the core 122 restrained from entering the throat 108 until appropriate in a number of ways, including without limitation a plate restraint 117 comprised of a sheet of material contacting the core 122 from the side, below or above (i.e. as shown in FIGS. 1, 2, 4–11) that helps restrain the core 122 due to the orientation and rigidity of the plate; a door that slides, swings, rotates, or rises into position in front of the core 122; a fence made up of a plurality of rods, pegs or plates that is oriented above, below, or beside the core 122 and slides, swings, rotates, rises up, or otherwise moves into position in front of the core 122; and any other barricading structure or device that helps restrain the core 122 from entering the throat 108 until the appropriate time. As illustrated in FIG. 4, the core 122, complete with adhesive in some embodiments, is trapped between the chain conveyor 115 and the plate restraint 117, located above the core 122 in the illustrated embodiments. A row of perforations 109 is shown just coming onto the first winding roll 106. The web separation bar 124, the conveyor 115 and the core inserter 111 are about to begin moving to initiate the separation and core insertion processes.

FIG. 5 shows that the log 112 has started to move away from the first winding roll 106, initiating the discharge process. This movement can be the result of slowing down the second winding roll 116 relative to the first winding roll 106, speeding up the first winding roll 106 relative to the second winding roll 116, or both. The web separation bar 124 has accelerated through 270° of rotation from rest (as shown in FIG. 4) to a tip velocity of 130% of the velocity of the web 102 adjacent the first winding roll 106. The perforation 109 has traveled around the first winding roll 106 to a position close to the core 122. The core inserter 111 is pushing the core 122 out from under the plate restraint 117, toward the throat 108 and onto the core support plates 110. In some embodiments, the core inserter 111 accelerates the core 122 to approximately 50% of the velocity of the web 102 adjacent the first winding roll 106. The core 122 then travels along the core support plates 110 at a velocity approximately 50% of the velocity of the web adjacent the first winding roll 106, as explained above.

FIG. 6 shows the log 112 continuing to move away from the first winding roll 106. The core 122 has been inserted between the first winding roll 106 and the core support plates 110, thereby forming the web control point 150 as explained above. The web separation finger tips 132 have passed through the core support plates 110 to an area between the

11

first winding roll 106 and the core support plates 110. The core inserter 111 and the web separation bar 124 have been timed relative to the perforation system 104 to place a single row of perforations 109 between the core 122 and web separation bar 124 adjacent the first winding roll 106.

FIG. 7 shows the log 112 moved away from the first winding roll 106 enough to allow the rider roll 118 to drop toward the second winding roll 116. The core 122 is driven by the first winding roll 106 and is rolling along the core support plates 110. The separation bar 124 is in contact with the web 102 adjacent the low friction surfaces 136 of the first winding roll 106. The web 102 at the web control point 152 is therefore rapidly accelerated to the velocity of the web separation bar 124. This acceleration of the web 102 causes the web 102 to become slack downstream of the web separation bar 124 and to become taut upstream of the web separation bar 124. Specifically, the web 102 stretches in the zone between the two web control points 150, 152, causing the web 102 to rupture into a leading edge 142 and a trailing edge 144 along the properly positioned row of perforations 109 located between the two web control points 150, 152.

FIG. 8 demonstrates the transfer of the leading edge 142 of the ruptured web 102 to the core 122. In some cases, and depending upon the speed of the core 122 and the distance between the core 122 and the leading edge 142 of the ruptured web 102, a short, controlled fold-back of the web 102 can be formed on the core 122.

FIG. 9 shows the web separator 125 moving out of the core path and out of the area between the core support plates 110 and the first winding roll 106. The core 122 is moving toward the winding nip 114 between the first winding roll 106, the second winding roll 116 and the rider roll 118. The rider roll 118 has dropped down close to the second winding roll 116.

FIG. 10 shows a later stage in the winding process, with the core 122 in contact with the first winding roll 106, the second winding roll 116 and the rider roll 118.

Finally, as shown in FIG. 11, the rider roll 118 begins to move upward as the new winding log 112' increases in diameter. The conveyor 115 has indexed a new core 122' into position for the next core insertion step. Winding can continue until the log 112' nears completion, at which time the above-described process can repeat, beginning as depicted in FIG. 4.

As best illustrated in FIGS. 1 and 2, a conveyor 115, a core inserter 111 and a plurality of core support plates 110 are used to insert and guide the cores 122 into the winding nip 114. However, cores 122 can instead be inserted and/or guided to a winding nip (e.g. a two- or three-roll winding nip) via other insertion devices that are well within the spirit and scope of the present invention. For example, one or more fingers or other protrusions can extend from a ring that, when rotated, picks up cores 122 and transports them toward the winding nip 114; a pulley system that transports cores to a location where a lever, pressurized air jet, vacuum system or other mechanism directs the core 122 into the winding nip 114; an elevating platform that brings cores 122 toward a desired position where a lever, pressurized air jet, vacuum system or other mechanism directs cores 122 into the winding nip 114; one or more ramps, rails, ducts, beds, gutters and the like that guide cores 122 to the winding nip 114 via gravity, a pressurized air jet, a vacuum system, or other mechanism; a series of valves within or along a ramp, rail, duct, bed, gutter and the like that indexes and advances cores 122 to the winding nip 114 by incorporating pushers or pressure gradients to force cores 122 through the valves

12

or timers to actuate the opening and closing of the valves to allow cores 122 to move through the valves at appropriate times; a rotatable or swinging arm that transports cores 122 to the winding nip 114; and any other inserting and guiding device or system known to those skilled in the art.

In some embodiments of the present invention, the core inserter 111 comprises one or more paddles that rotate about an axis T to push the core 122 out from under the plate restraint 117 and into the throat 108 as shown in FIGS. 1, 2 and 4-13. However, in other embodiments, the core inserter 111 does not rotate about an axis, but rather follows the conveying motion of the conveyor 115, moves along an arcuate path independent from but adjacent the conveyor 115, moves along a straight path independent from but adjacent the conveyor 115, follows an aperture in a cam member, and/or follows any other path or moves in any other manner for moving the core 122 as described above. In some embodiments, the core inserter 111 is comprised of one or more rods, plates, fingers and/or any other element capable of pushing the core 122 into the throat 108. In other embodiments, the core inserter 111 has one or more curved or bowed surfaces, is spherical, or has a cross-section that is trapezoidal, triangular, round, diamond-shaped, or has any other shape or cross-sectional shape. In still other embodiments, the core inserter 111 does not push the core 122 by contacting the core 122 along a longer edge of the core inserter 111, but rather pushes the core 122 into the throat 108 by poking the core 122 with a shorter end of the inserter 111, or pushes the core 122 in any other manner known to those skilled in the art. Accordingly, these and any other devices or structures capable of transporting and inserting cores 122 into the winding nip 114 can be employed without departing from the present invention. However, regardless of the device or system that transports cores 122 to the winding nip 114, the web separation bar 124 of the present invention can still be employed as described above to separate the web downstream of the core 122 being inserted.

The core support surface 110, if employed, can be any surface along which cores 122 can be guided toward the winding nip 114. For example, the core support surface 110 can be defined by one or more sides, edges or other surfaces, of one or more plates, rods, bars or other elements extending any distance past and/or around the first winding roll, can be a sheet of material, a grid or a mesh structure, a frame of multiple elements and the like. The core support surface 110 illustrated in FIGS. 1-13 is curved, but the core support surface 110 can have a number of different forms, including without limitation flat, semicircular, and any form capable of transporting cores 122 to the web 102. In some embodiments, the core support surface 110 is comprised of a plurality of rods with rectangular cross-sections, but the core support surface 110 can be a number of different shapes; for example, the core support surface 110 can be a solid sheet or plurality of rods, bars, plates or other elements with an ellipsoidal shape, square cross-section, circular cross-section, triangular cross-section, trapezoidal cross-section, and any other shape or cross-section known to those skilled in the art. In the embodiments illustrated in FIGS. 1-13, the core support surface 110 is stationary, but the core support surface 110 can be movable; that is, movable only when actuated, movable by rotation, movable by swinging about a hinge, movable by sliding along a straight or arcuate path, and movable by any other devices known to those skilled in the art. In still other embodiments, the core support surface 110 can be comprised of a plurality of rods transversely spaced an equal distance apart; however, the core

support surface **110** can be comprised of a plurality of sheets, plates, rods, bars or other elements and can have a number of different schemes for spacing these elements; for example, the elements can be spaced longitudinally, transversely, equally, unequally, randomly, and follow any other scheme or pattern of spacing without departing from the present invention. In yet other embodiments, the core support surface **110** is comprised of a plurality of rods oriented longitudinally (as shown in FIGS. 1–13 and especially FIG. 3), but the core support surface **110** can be oriented with respect to the advancing web in a number of ways, including without limitation being oriented longitudinally, transversely, partially transversely and partially longitudinally, radially, and be oriented in any other manner known to those skilled in the art. In short, the core support surface **110** can comprise any other surface or plurality of surfaces capable of guiding cores **122** toward the winding nip **114**.

Although in the embodiment illustrated in FIGS. 1–11, the web separator **125** is elongated, rotatable about an axis and comprised of a plurality of web separation fingers **130** with resilient tips **132**, the web separator **125** can take a number of different forms while still being movable toward the web **102** at a velocity at least equal to that of a portion of the web **102** adjacent the web separator **125** at the time of separation for the purpose of contacting and separating the web **102**. For example, the web separator **125** comprising one or more web separation fingers **130**, bases **133**, and/or tips **132** can be mounted to a linear actuator **154** for movement toward the web **102** along a linear path (as shown in FIG. 12), or mounted to a conveying belt **156** equipped with one or more web separation fingers **130**, paddles, or other protrusions, bases **133** and/or tips **132** (as shown in FIG. 13). Many types of linear actuators can be employed, including without limitation a solenoid, hydraulic or pneumatic cylinder, magnetic rail, and the like. In the embodiment shown in FIG. 12, the linear actuator **154** is oriented at an angle of approximately 30° with respect to the advancing stream of web **102**; however, the linear actuator **154** can be oriented at any possible angle with respect to the web **102** so as to contact and separate the web **102**. In the embodiment shown in FIG. 13, the web separator **125** is mounted at 90° with respect to a rectangular-shaped conveying belt **156**. However, in other embodiments, the web separator **125** can be mounted to the conveying belt **156** at any angle possible and capable of moving along any path possible as defined by the conveying belt **156**, such as triangular, circular and other paths as described above. In other embodiments, the web separator **125** can move toward the web **102** via a combination of devices or actuators, including without limitation the aforementioned devices and actuators.

The embodiments illustrated in FIGS. 1–13 and especially FIGS. 1–3 and 6–8, show the web separator **125** contacting the web **102** against a surface of the first winding roll **106**. However, in some embodiments, the web separator **125** presses the web **102** against one or more fingers, plates, spheres, and any other elements against which the web separator **125** can press instead of or in addition to pressing the web **102** against the roll **106**.

The embodiment best illustrated in FIG. 3 shows the web separator **125** comprising fingers **130**, bases **133**, and tips **132** having recesses **138**. However, the web separator **125** need not be comprised of a plurality of web separation fingers **130** with resilient tips **132** (whether or not having recesses **138** in the tips to accommodate the high friction surfaces **134** of the first winding roll **106**). Furthermore, the web separator **125** can make minor or brief contact with the

web **102** sufficient to accelerate the web **102** to the breaking point, without the web **102** being required to slip on a first winding roll **106** in order to separate. In some embodiments, the web separator **125** can instead comprise a plurality of sharp web separation fingers **130**, elongated or otherwise, for extension into grooves in the first winding roll **106**. In other embodiments, the sharp or otherwise web separation fingers **130** can extend into grooves in any surface adjacent the advancing web **102**, whether a winding roll, belt or other surface capable of advancing or supporting the web **102**. In still other embodiments, the web separation fingers **130** can be one or more bars, rods, plates, or other elements that press the web **102** against a stationary or moving surface. In yet other embodiments, the sharp or otherwise web separation fingers **130** can merely extend toward, into or through the advancing web **102** to separate the web **102**, whether perforated or not, without the use of any first winding surface **106**. Such web separation fingers **130** can be sharp or can otherwise act as blades against the web **102** and/or first winding surface **106** in order to cut the web **102**, if desired. Additionally, in still other embodiments, the first winding roll **106** can be equipped with rotating blades or protrusions that move toward the web **102** at a velocity at least equal to that of the web **102** to engage the fingers **130** of the web separator **125**, the fingers **130** functioning as anvils. Alternatively, the blades of the first winding roll **106** can run near or adjacent the advancing web **102**, and the fingers **130** functioning as anvils can move toward the web to contact the blades of the first winding roll **106** at a velocity at least equal to that of the advancing web **102**.

Furthermore, the resilient tip **132** of the web separation finger **130** need not rotate or follow a circular path to contact and separate the web **102**, but can follow one or more of a number of different paths, as explained above. The web separator **125** can follow any possible path as long as the web separator **125** is movable toward and away from an advancing stream of web at a velocity at least equal to that of the web **102** adjacent the web separator **125** at the time of web separation in order to separate the web **102**.

A number of alternative elements and structures can be employed for this purpose. By way of example only, the web separator **125** can comprise a roll adjacent the first winding roll **106** and rotatable about an axis at a speed greater than that of the advancing stream adjacent the web separator **125**. Such a roll can be moved in any conventional manner toward the advancing stream of web **102** to separate the web **102**. If desired, this roll can comprise one or more strips of resilient or rigid material of high or low friction extending transversely or longitudinally along the roll, or can have a continuous outer surface composed of a resilient or rigid material of high or low friction. In embodiments where the core support surface **110** and first winding roll **106** as depicted in FIGS. 1–13 are employed, this web separation roll **125** can instead be a cylindrical eccentric roll having grooves defining portions of the roll that can pass through the core support surface **110** to contact the web **102**. In still other embodiments, the web separator **125** can be a moving belt or wheel with paddles or fingers, or other types of protrusion extending into contact with the web **102** as needed.

In those embodiments in which a core support surface **110** and a web separator **125** are employed, these two devices do not necessarily need to cooperate (i.e. interdigitate; contact one another; move near, past, or through each other; or operate synchronously). These and any other structure capable of separating the web **102** by moving toward the web **102** at a velocity at least equal to that of a portion of the

15

web 102 adjacent the first winding roll 106 can be employed as alternatives for the web separator 125 and, thus, can be employed without departing from the present invention.

The rolls described above can have a number of different structures, as stated above, including without limitation 5 belts, wheels, stationary surfaces, stationary tracks having a plurality of rollers or wheels for conveying material, and any other conveying or supporting structure that performs the function of transporting, supporting, and/or winding the web 102. In some embodiments, the first winding surface 106 has a plurality of alternating longitudinal strips of high friction surfaces 134 and low friction surfaces 136; however, this need not be the case, but rather the first winding surface 106 can have one continuous outer surface of high or low friction including without limitation steel; aluminum; poly (tetrafluoroethylene) (PTFE; Teflon®); rubber; emery cloth; 10 wood, natural or otherwise; ultra-high molecular weight poly(ethylene); silicone; and any other surface capable of acting as at least an outer layer on the first winding surface 106 for transporting, supporting and/or winding the web 102. The first winding surface 106 need not transport the web necessarily, but, if employed, provides a surface against which the web separator 125 can press the web 102 for the purpose of separating the web 102. Alternatively, the web 102 can move through the winding area 101 without being directly adjacent any winding surface, in which case the tension in the web 102 is selected to be sufficient for a web separator 125 approaching, contacting and pulling the web 102 at a velocity at least equal to that of the running speed of the web 102 to separate the web 102. Additionally, even if a first winding surface 106 is employed for advancing the web 102, the web separator 125 need not cooperate (i.e. contact; move near, past or through; interdigitate; or operate synchronously) with this surface 106 in order to separate the web 102. Thus, the above and any other structures capable of transporting and winding the web 102 are considered to fall within the spirit and scope of the present invention.

What is claimed is:

1. A method of winding a web onto a core in a rewinder, the rewinder winding a web of material adjacent at least one of a first winding roll, a second winding roll and a rider roll, at least two of the first, second, and rider rolls defining a winding nip in the rewinder, the method comprising:

moving the core onto at least one core support plate and toward the web running into the winding nip;

moving a web separation bar toward the web at a velocity at least equal to that of a portion of the web adjacent the first winding roll;

accelerating the web separation bar to a velocity at least 125% of that of the web;

pressing the web between the web separation bar and a surface on an opposite side of the web;

separating the web into a leading edge and a trailing edge; moving the web separation bar away from the web; and winding the leading edge around the core.

2. The method as claimed in claim 1, wherein pressing the web includes pressing the web between the web separation bar and a surface of the first winding roll.

3. The method as claimed in claim 1, wherein pressing the web includes pressing the web between the web separation bar and a surface located between the first winding roll and the web.

4. The method as claimed in claim 1, wherein moving the web separation bar toward the web includes moving the web separation bar via a linear actuator.

5. The method as claimed in claim 1, wherein moving the web separation bar toward the web includes moving the web separation bar via a conveying belt.

16

6. The method as claimed in claim 1, further comprising perforating the web.

7. The method as claimed in claim 1, further comprising accelerating the web at the web separation bar by contact of the web separation bar against the web.

8. The method as claimed in claim 1, further comprising increasing tension in the web upstream of the web separation bar.

9. The method as claimed in claim 1, further comprising separating the web upstream of the web separation bar.

10. The method as claimed in claim 1, wherein a first point of contact is defined between the first winding roll and the core, and a second point of contact located downstream of the first point of contact is defined between the first winding roll and the web separation bar; the method further comprising increasing tension in the web between the first and second web contact points.

11. The method as claimed in claim 10, further comprising separating the web between the first and second web contact points.

12. The method as claimed in claim 1, further comprising moving the core off of the core support plate to roll upon the second winding roll.

13. The method as claimed in claim 1, wherein the web separation bar comprises a plurality of tips; and moving the web separation bar includes moving the plurality of tips adjacent the at least one core support plate.

14. The method as claimed in claim 1, further comprising accelerating the web separation bar to a velocity at least equal to that of the web.

15. The method as claimed in claim 1, further comprising accelerating the web separation bar to a velocity at least 150% of that of the web.

16. The method as claimed in claim 1, further comprising moving a web separator defined at least in part by the web separation bar toward the web at a velocity at least equal to that of the web.

17. A method of winding a web onto a core in a web rewinder, including a winding nip defined at least partially by a first winding roll, the method comprising:

passing the web over a surface of the first winding roll; moving the core adjacent the web along at least one core support plate adjacent the first winding roll;

moving a web separator toward the web at a velocity at least equal to that of a portion of the web adjacent the first winding roll;

accelerating the web separator to a velocity at least 125% of that of the web;

pressing the web between the web separator and the first winding roll; and separating the web.

18. The method as claimed in claim 17, wherein moving the web separator toward the web includes moving the web separator via a linear actuator.

19. The method as claimed in claim 17, wherein moving the web separator toward the web includes moving the web separator via a conveying belt.

20. The method as claimed in claim 17, further comprising releasing the web compressed between the web separator and the first winding roll.

21. The method as claimed in claim 17, further comprising winding the web onto the core as the core moves toward the winding nip.

22. The method as claimed in claim 17, further comprising moving the core off of the at least one core support plate for continued winding of the web onto the core in the winding nip.

17

23. The method of claim 17, further comprising perforating the web.

24. The method as claimed in claim 17, further comprising moving the core onto the at least one core support plate substantially simultaneously to moving the web separator toward the web.

25. The method as claimed in claim 17, further comprising winding the web in a winding nip, defined by a first winding roll, a second winding roll and a rider roll.

26. The method as claimed in claim 17, wherein the web separator has a plurality of fingers, bases and tips; the method further comprising moving the fingers between and through the at least one core support plate as the web separator approaches the web.

27. The method as claimed in claim 17, further comprising accelerating the web separator to a velocity at least equal to that of the web.

28. The method as claimed in claim 17, further comprising accelerating the web separator to a velocity at least 150% of that of the web.

29. The method as claimed in claim 17, further comprising accelerating the web at the web separator by contact between the web separator and the web.

30. The method as claimed in claim 17, further comprising increasing tension in the web upstream of the web separator by contact of the web with the web separator.

31. The method as claimed in claim 17, further comprising separating the web up stream of the web separator.

32. The method as claimed in claim 17, wherein a point of contact is defined as the point where the web separation bar contacts the first winding roll; the method further comprising increasing tension in the web upstream of the web contact point.

33. The method as claimed in claim 32, further comprising separating the web upstream of the point of contact.

34. An apparatus capable of winding a web onto a core in a winding nip, the apparatus comprising:

a first winding roll at least partially defining the winding nip, the web running adjacent the first winding roll;

a core support plate on which the core is received and moved toward the winding nip; and

a web separation bar adjacent the first winding roll, the web separation bar movable in a non-transverse direction to the web into contact with the web at a velocity at least equal to that of the web, wherein the web separation bar is movable into and out of pressing relationship with the web via the use of a conveying belt.

35. The apparatus as claimed in claim 34, wherein the core support plate is one of a plurality of core support plates onto which the core is guided toward the web.

36. The apparatus as claimed in claim 34, wherein the core support plate is curved.

37. The apparatus as claimed in claim 34, wherein the web separation bar is comprised of a plurality of tips which are movable to contact the web.

38. The apparatus as claimed in claim 37, wherein the tips are composed of a deformable and resilient material.

39. The apparatus as claimed in claim 37, wherein the first winding roll has alternating annular rings of high and low friction surfaces.

40. The apparatus as claimed in claim 37, wherein the tips have recesses.

41. The apparatus as claimed in claim 34, wherein the winding nip is defined by a first winding roll and a second winding roll.

42. The apparatus as claimed in claim 34, wherein the winding nip is defined by a first winding roll, a second winding roll and a rider roll.

18

43. The apparatus as claimed in claim 34, further comprising a web separator that is defined at least in part by the web separation bar.

44. The apparatus as claimed in claim 43, wherein the web separator has a plurality of fingers, bases and tips.

45. The apparatus as claimed in claim 34, wherein the web separation bar is rotatable into and out of pressing relationship with the web.

46. The apparatus as claimed in claim 34, wherein the web separation bar is movable into and out of pressing relationship with the web via the use of a linear actuator.

47. The apparatus as claimed in claim 34, wherein the web separation bar is movable to accelerate to a substantially similar velocity as that of the web.

48. The apparatus as claimed in claim 34, wherein the web separation bar is movable to accelerate to a velocity at least equal to that of the web.

49. The apparatus as claimed in claim 34, wherein the web separation bar is movable to accelerate to a velocity at least 125% of that of the web.

50. The apparatus as claimed in claim 34, wherein the web separation bar is movable to accelerate to a velocity at least 150% of that of the web.

51. A rewinding apparatus for winding a web, comprising:
a first winding surface that transports and supports the web;

a core support surface on which cores are guided adjacent the first winding surface toward the web; and

a web separator adjacent the first winding surface and movable into and out of pressing relationship with the web at a velocity at least equal to that of the web, wherein the web separator is movable to contact the web via a linear actuator.

52. The apparatus as claimed in claim 51, wherein the core support surface is comprised of a plurality of plates.

53. The apparatus as claimed in claim 51, wherein the core support surface is curved.

54. The apparatus as claimed in claim 51, wherein the first winding surface is a roll.

55. The apparatus as claimed in claim 51, wherein the first winding surface is a conveying belt.

56. The apparatus as claimed in claim 51, wherein the web separator has a plurality of bases, fingers and tips.

57. The apparatus as claimed in claim 51, wherein the web separator is defined at least in part by a web separation bar.

58. The apparatus as claimed in claim 57, wherein the web separation bar is defined at least in part by a plurality of tips.

59. The apparatus as claimed in claim 58, wherein the tips are composed of a deformable and resilient material.

60. The apparatus as claimed in claim 51, wherein the web separator is rotatable toward and away from the web.

61. The apparatus as claimed in claim 51, wherein the web separator is movable to accelerate to a velocity at least equal to that of the web.

62. The apparatus as claimed in claim 51, wherein the web separator is movable toward the web at a velocity at least 125% of that of the web.

63. The apparatus as claimed in claim 51, wherein the web separator is movable toward the web at a velocity at least 150% of that of the web.

64. The apparatus as claimed in claim 51, wherein the web separator is movable to contact the web via a conveying belt.