US006135373A

United States Patent [19]

Davenport

[11] **Patent Number: 6,135,373**

[45] **Date of Patent: Oct. 24, 2000**

[54] **ROTARY GRINDER**

[76] Inventor: **Ricky W. Davenport**, P.O. Box 52154, Lafayette, La. 70505

[21] Appl. No.: **09/363,104**

[22] Filed: **Jul. 29, 1999**

Related U.S. Application Data

[62] Division of application No. 09/023,051, Feb. 13, 1998, Pat. No. 5,971,307, which is a continuation of application No. 08/802,848, Feb. 19, 1997, which is a continuation of application No. 08/477,229, Jun. 7, 1995, abandoned, which is a division of application No. 08/368,386, Dec. 30, 1994, Pat. No. 5,495,986, which is a continuation of application No. 08/060,753, May 12, 1993, abandoned.

[51] **Int. Cl.**[7] **B02C 7/16**

[52] **U.S. Cl.** **241/30**; 241/101.2; 241/261.2

[58] **Field of Search** 241/101.2, 21, 241/261.2, 261.3, 30

[56] **References Cited**

U.S. PATENT DOCUMENTS

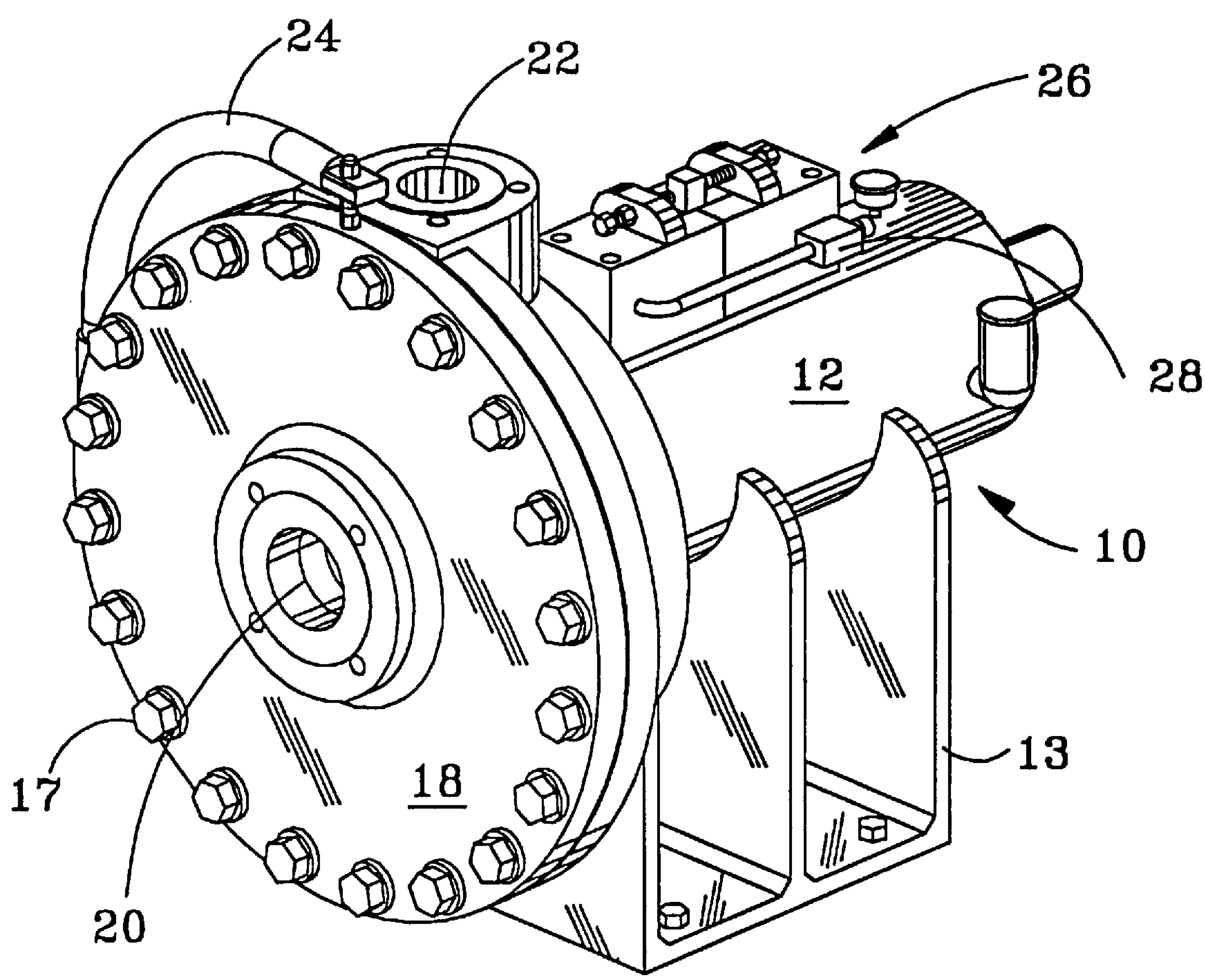
4,767,065 8/1988 Wray 241/261.3

Primary Examiner—Mark Rosenbaum
Attorney, Agent, or Firm—Robert N. Montgomery

[57] **ABSTRACT**

An in-line grinder has been developed which can be configured to perform in a variety of applications through the use of an adjustable rotor/stator assembly, removable shear bar, and a variety of interchangeable stator-rotor configurations. A unique drive system utilizing a mechanical seal cartridge provides maximum sealing with a minimum of shaft deflection and run-out thereby improving performance. These improvements collectively allow the grinder to be configured for optimum sizing of solids to a predetermined particle size for a broad range of materials. It has been demonstrated that a class of in-line grinders such as that described herein is applicable for sizing drill cuttings for injection into a subsurface formation by way of an annular space formed in a wellbore. The cuttings are removed from the drilling fluid, conveyed to a shearing and grinding system that converts the cuttings into a viscous slurry with the addition of water and viscosity enhancing polymers. The system in its simplest form comprises a slurry tank, a pump, and the instant in-line grinder. The pump circulates the mixture of cuttings and water (sea water) between the slurry tank and the in-line grinder. The ground mixture leaving the in-line grinder is then routed to an injection pump for high pressure injection into the formation.

8 Claims, 19 Drawing Sheets



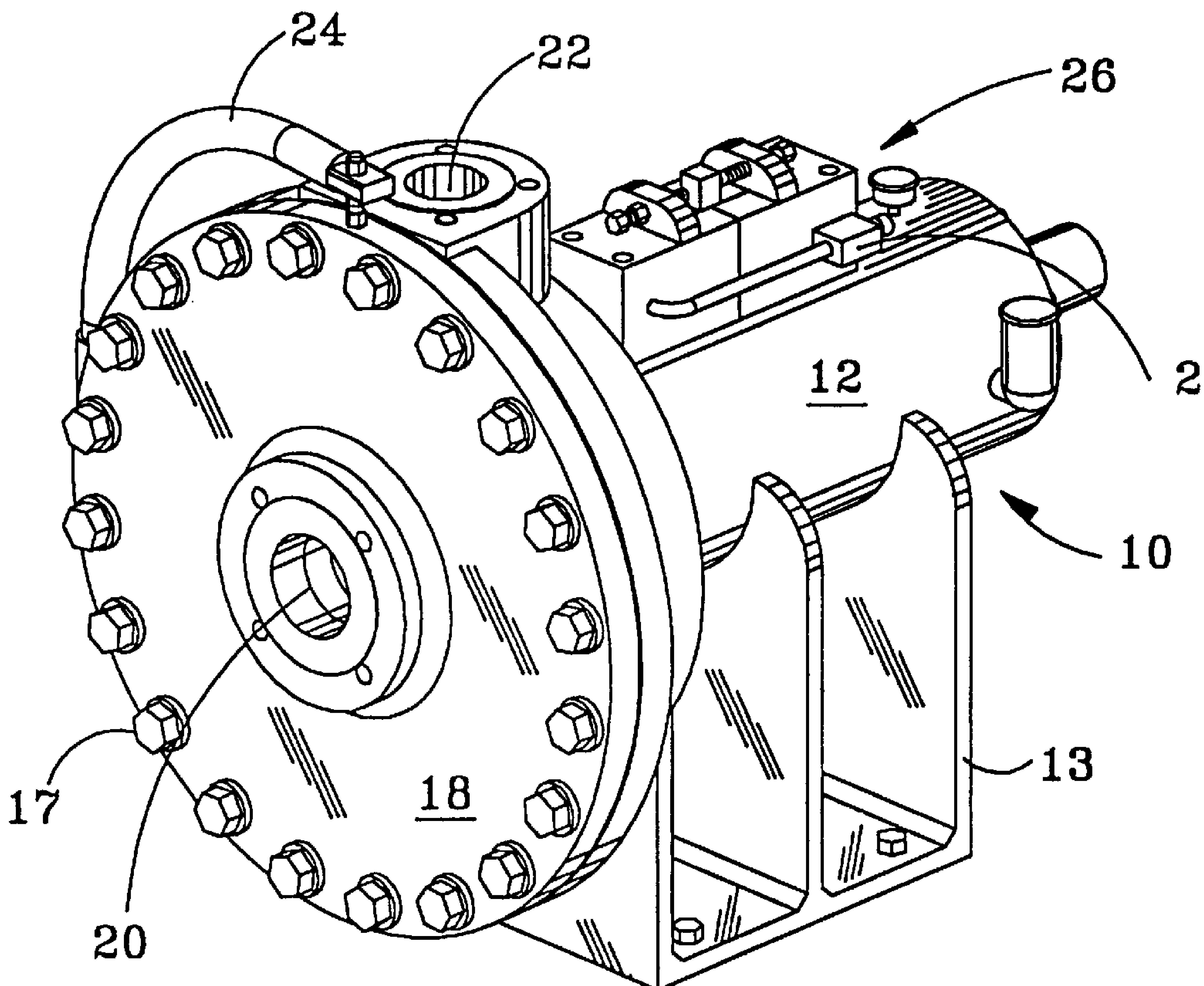


FIG. 1

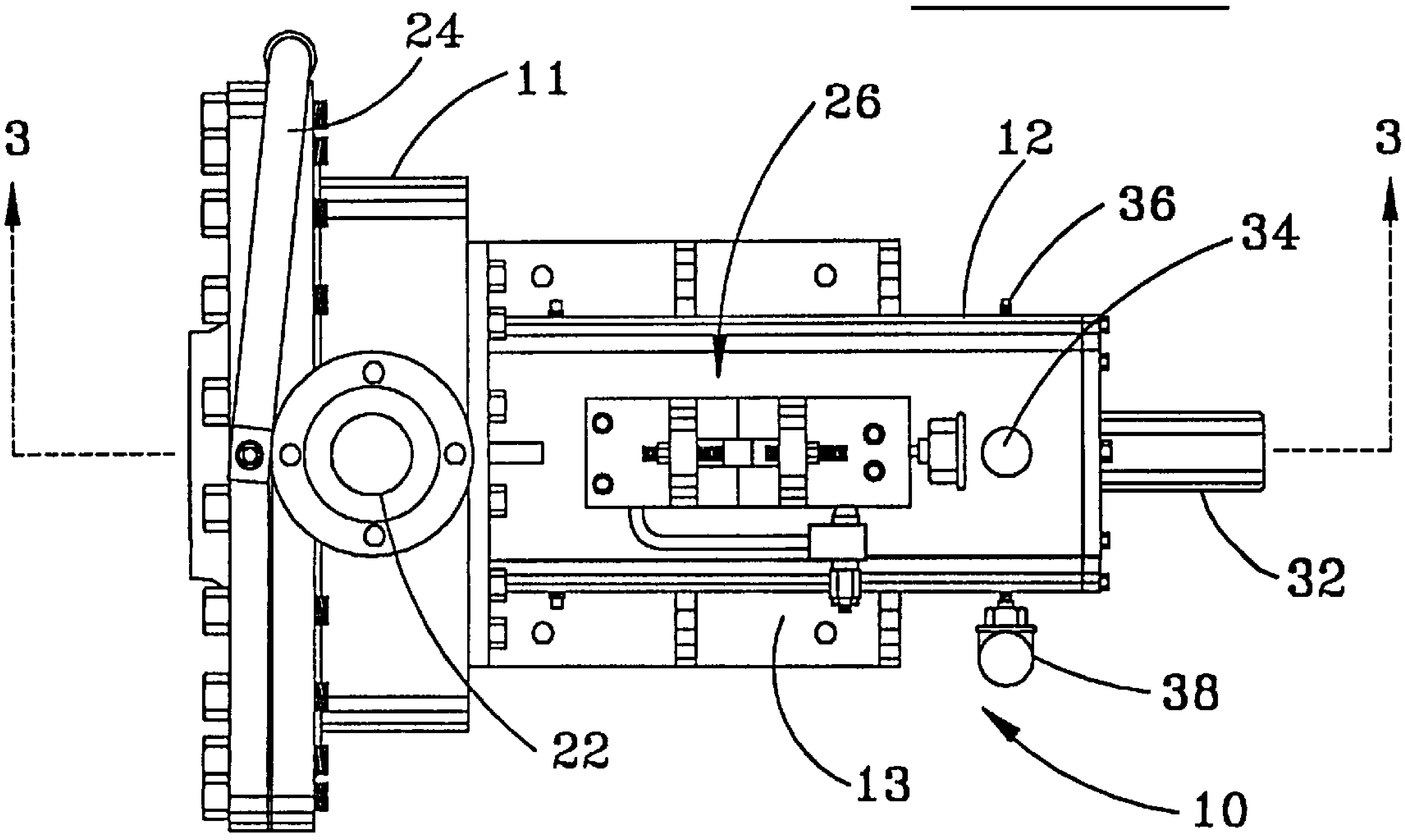


FIG. 2

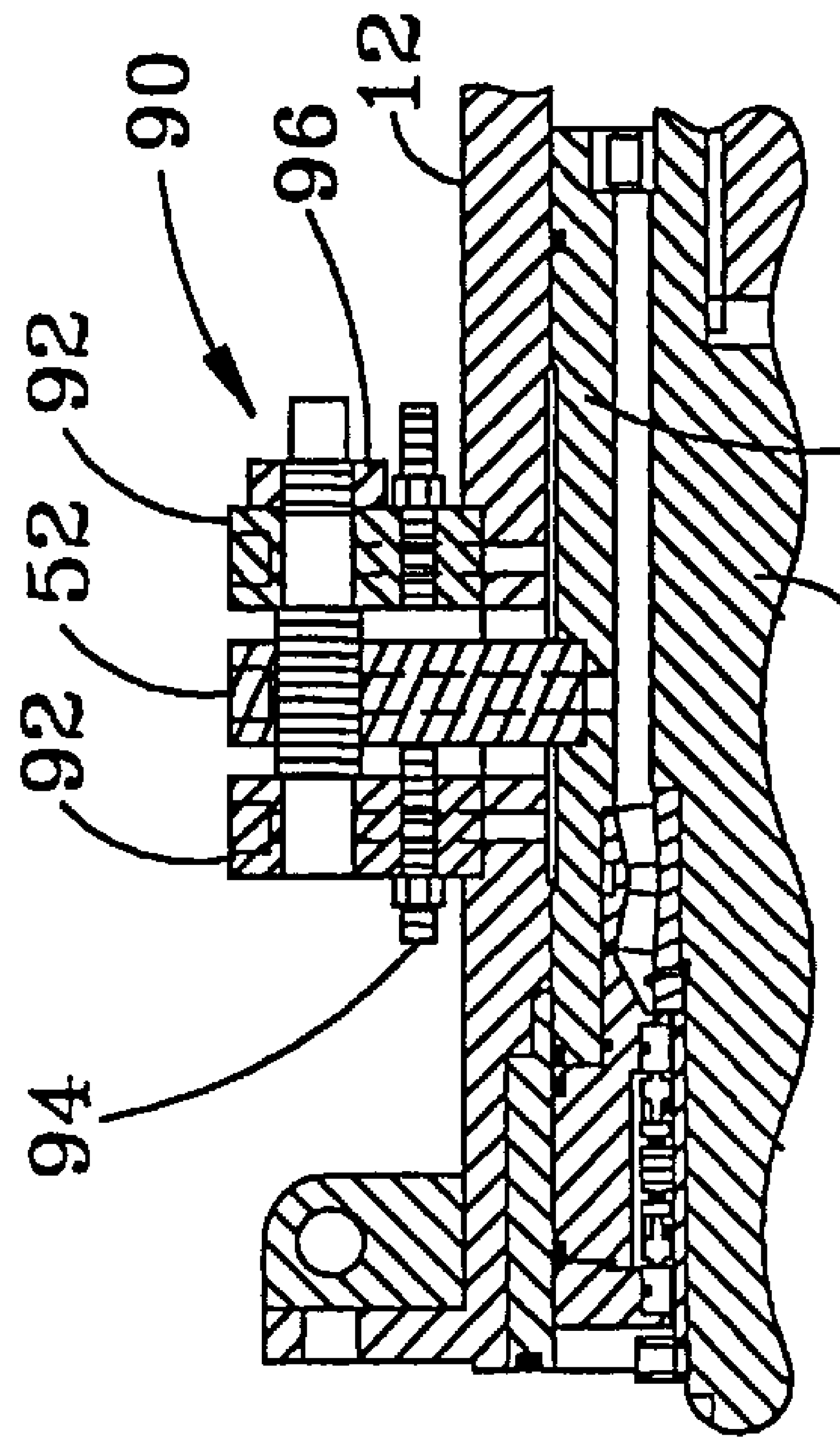

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FIG. 4

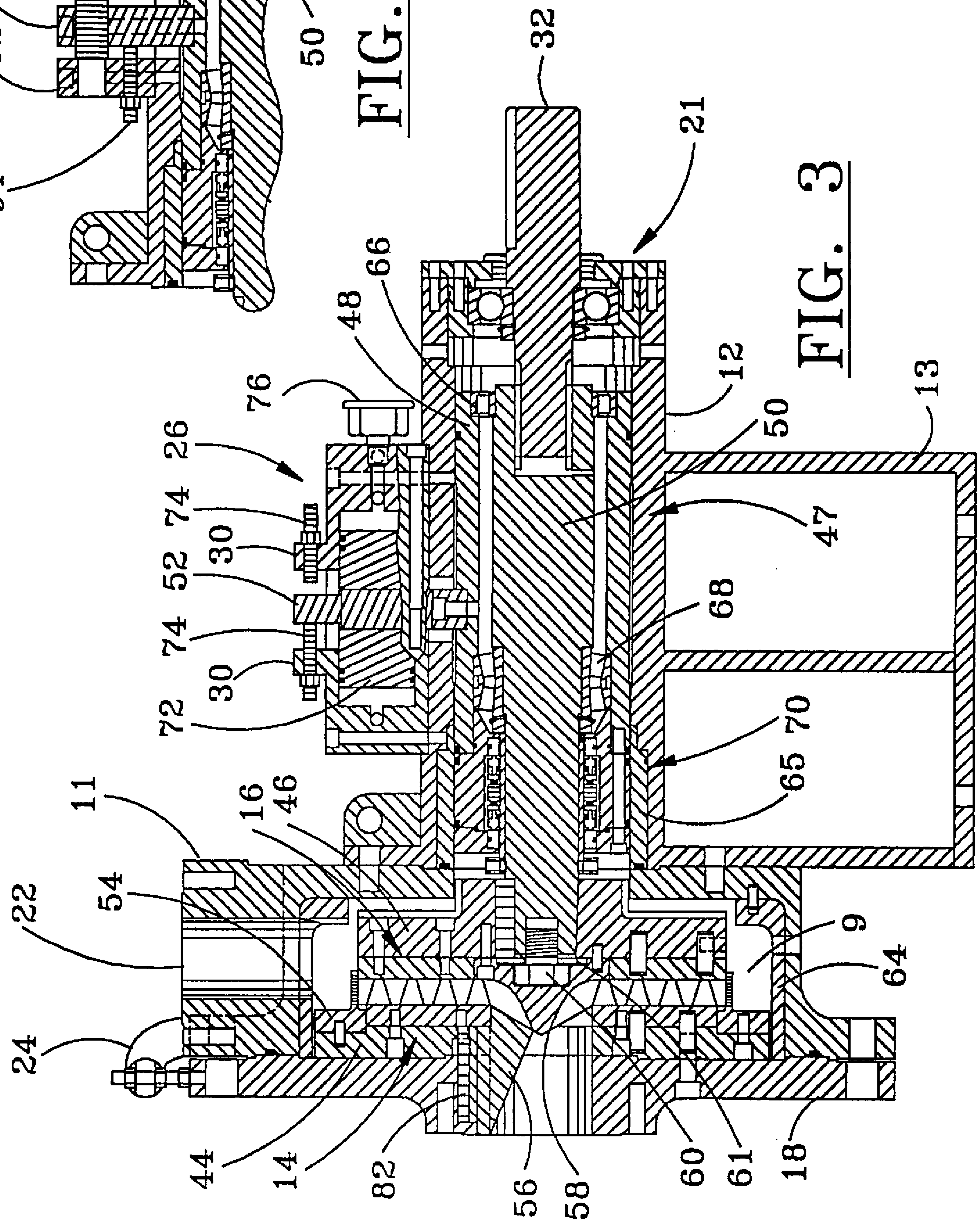
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FIG. 3

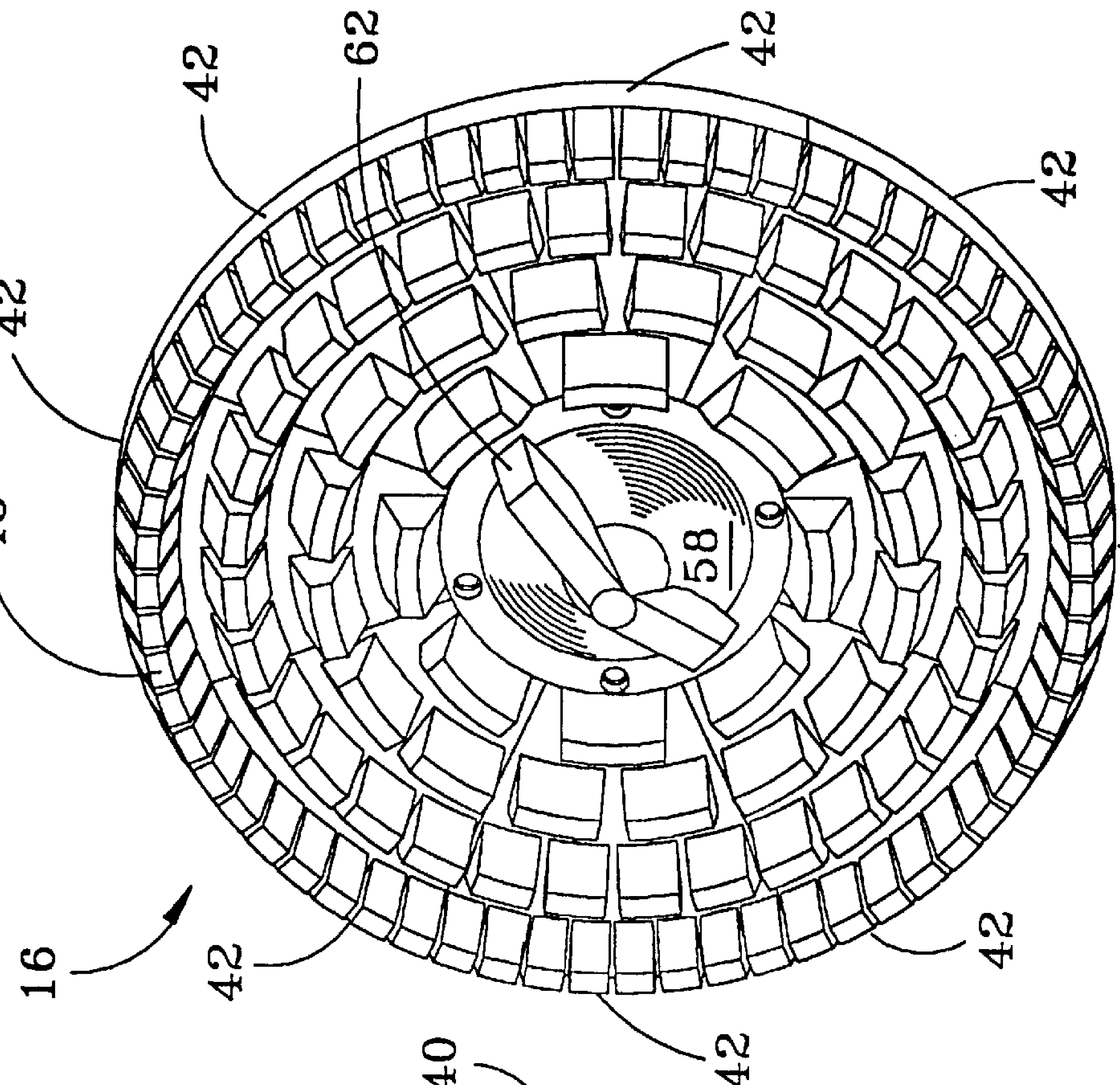

FIG. 6
Prior Art

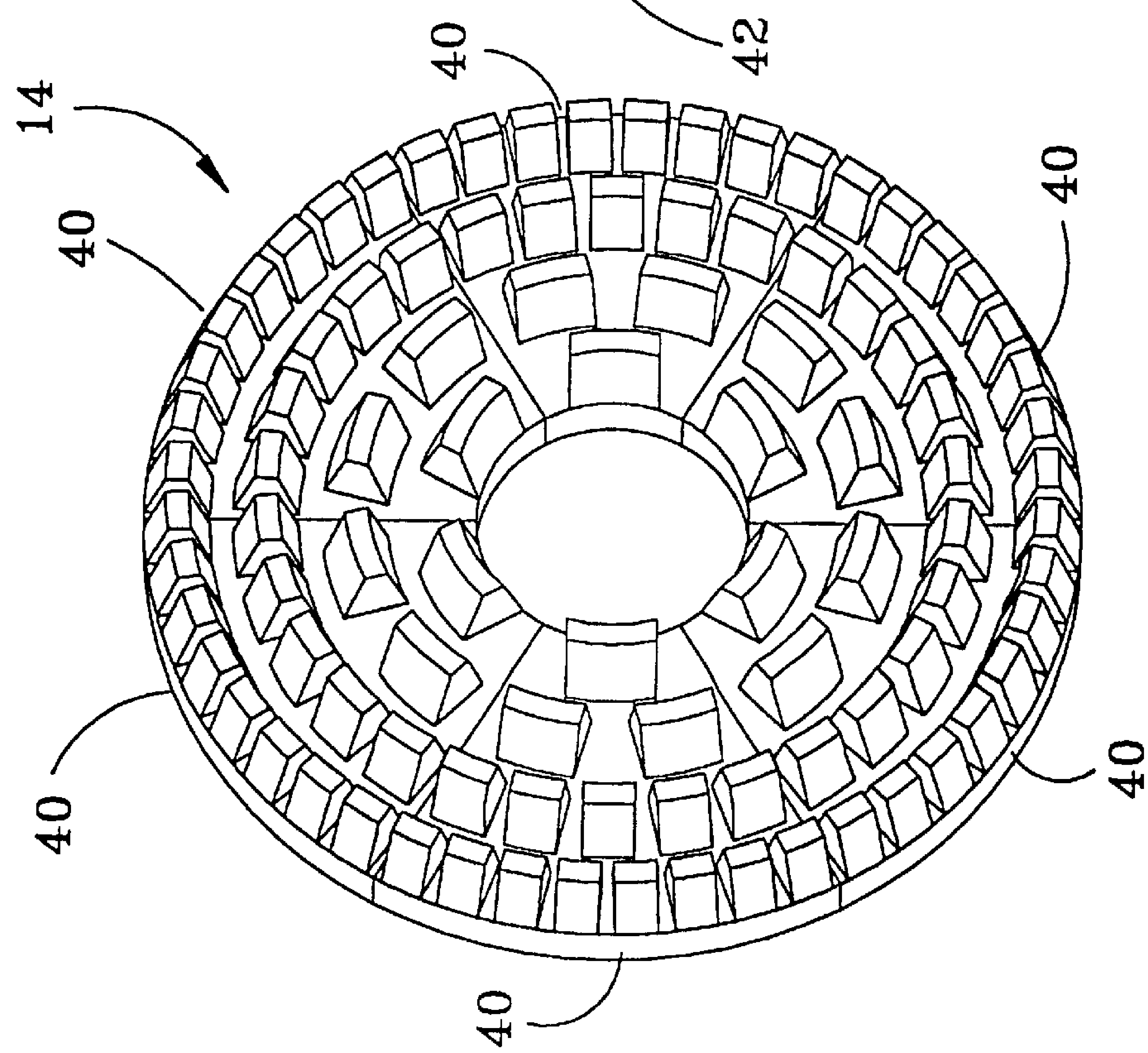

FIG. 5
Prior Art

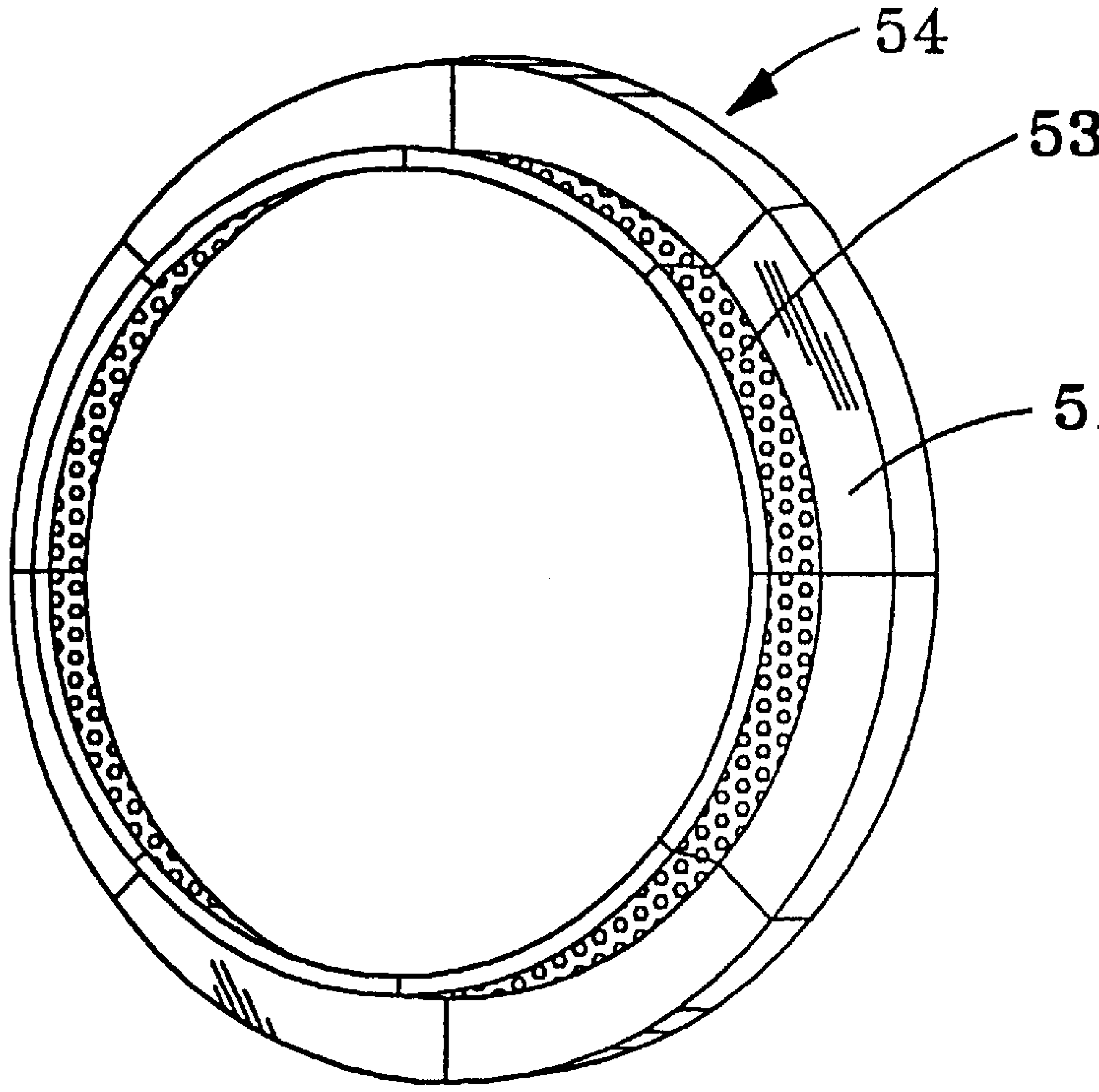

FIG. 7

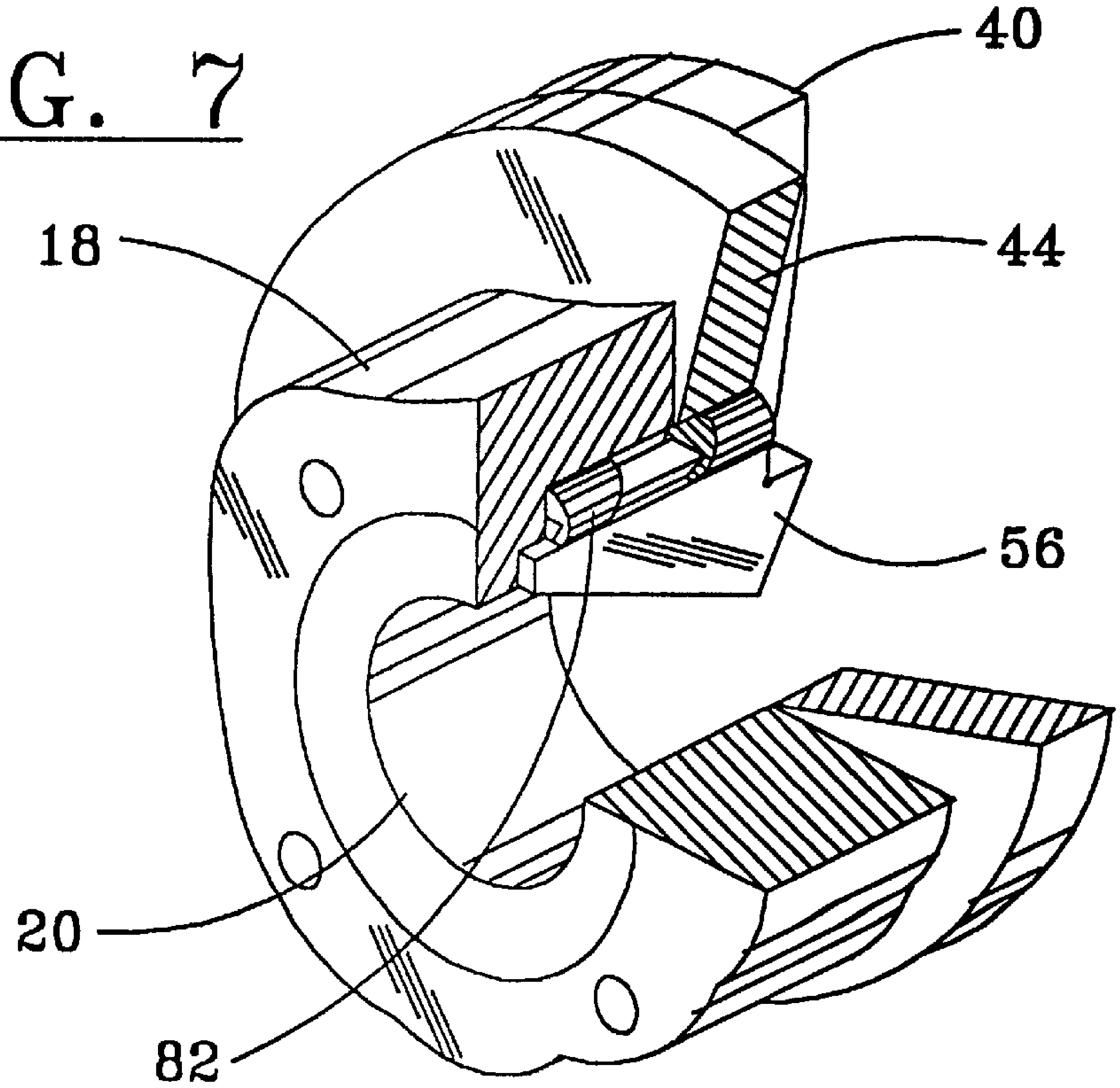

FIG. 8

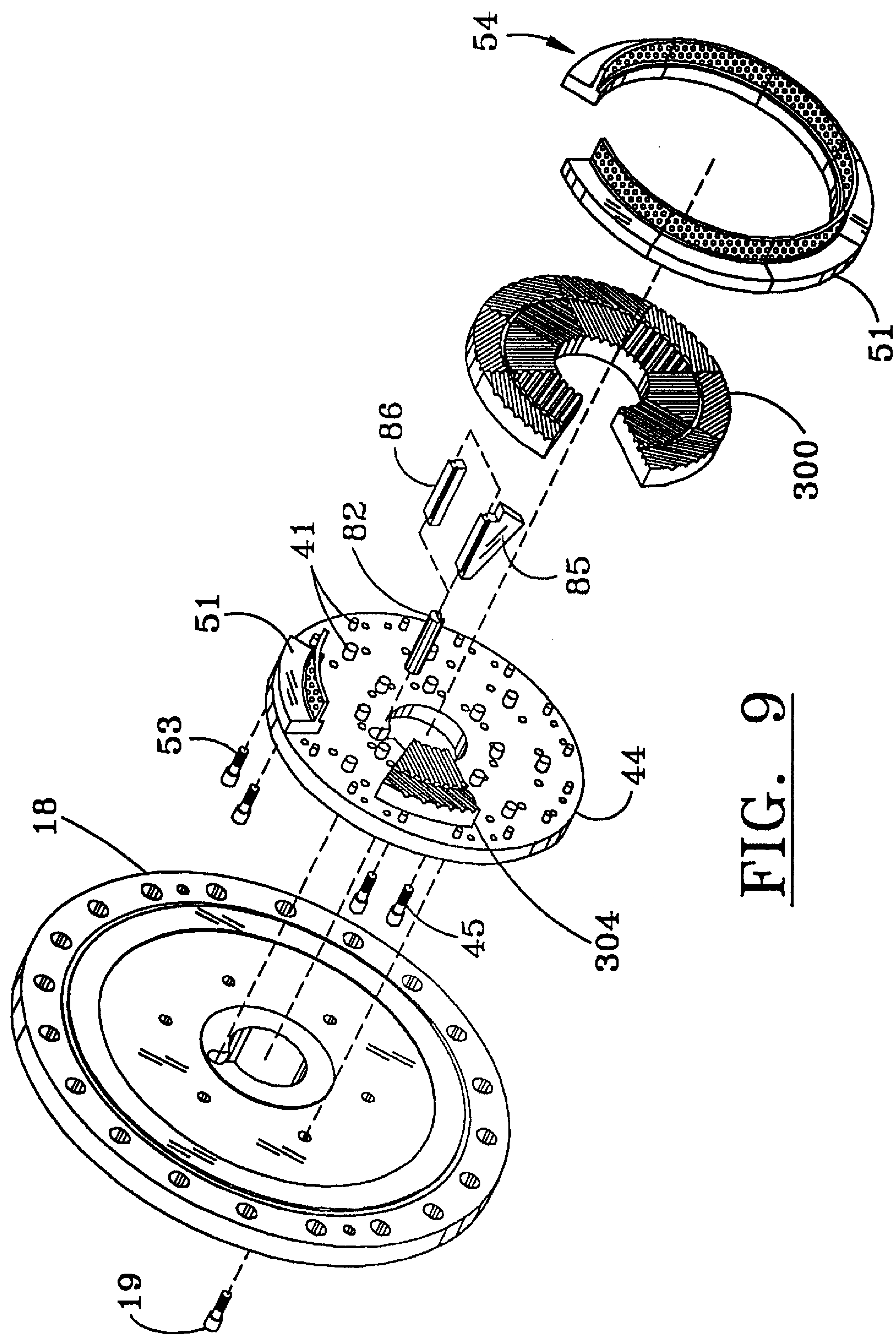

FIG. 9

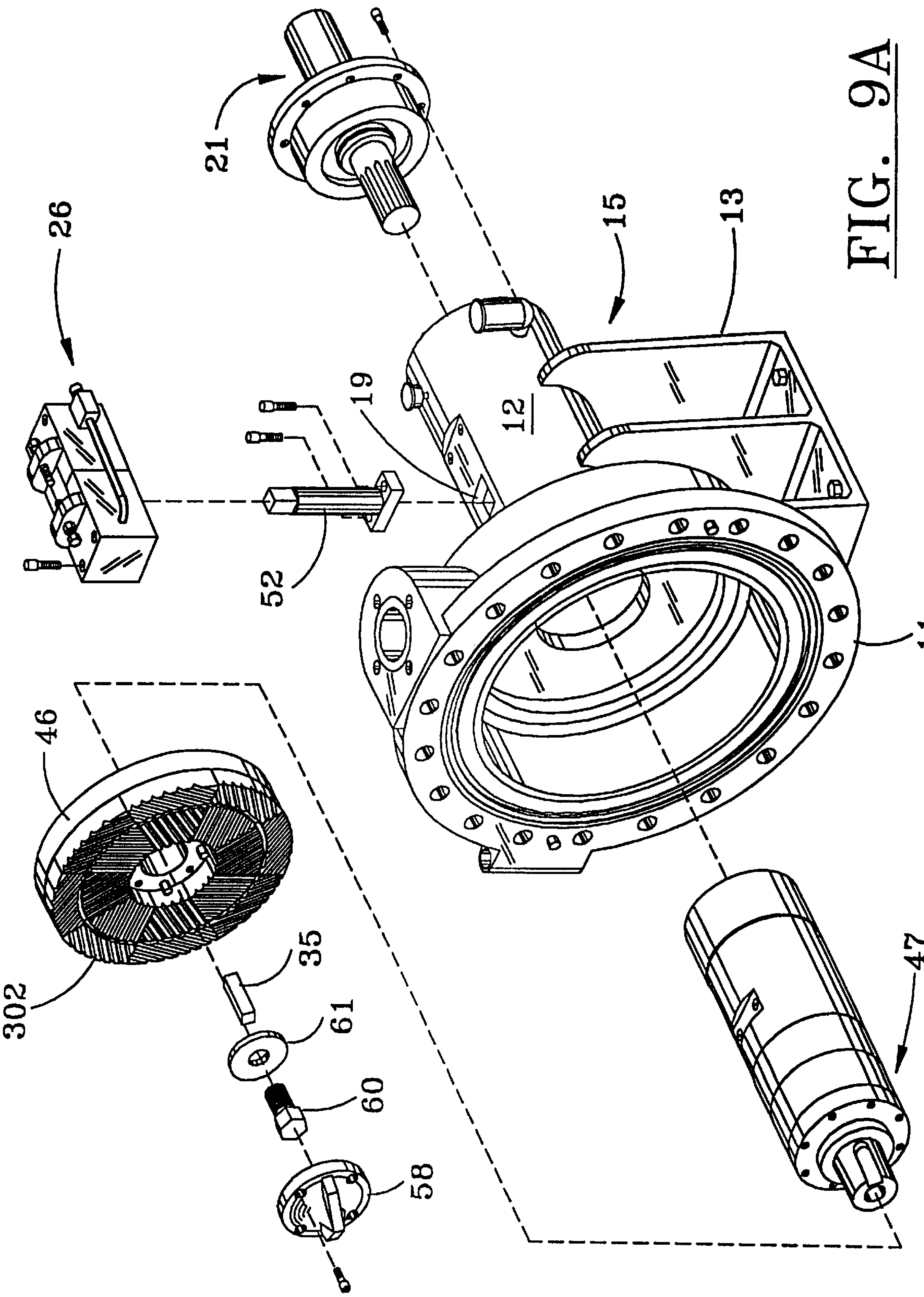

FIG. 9A

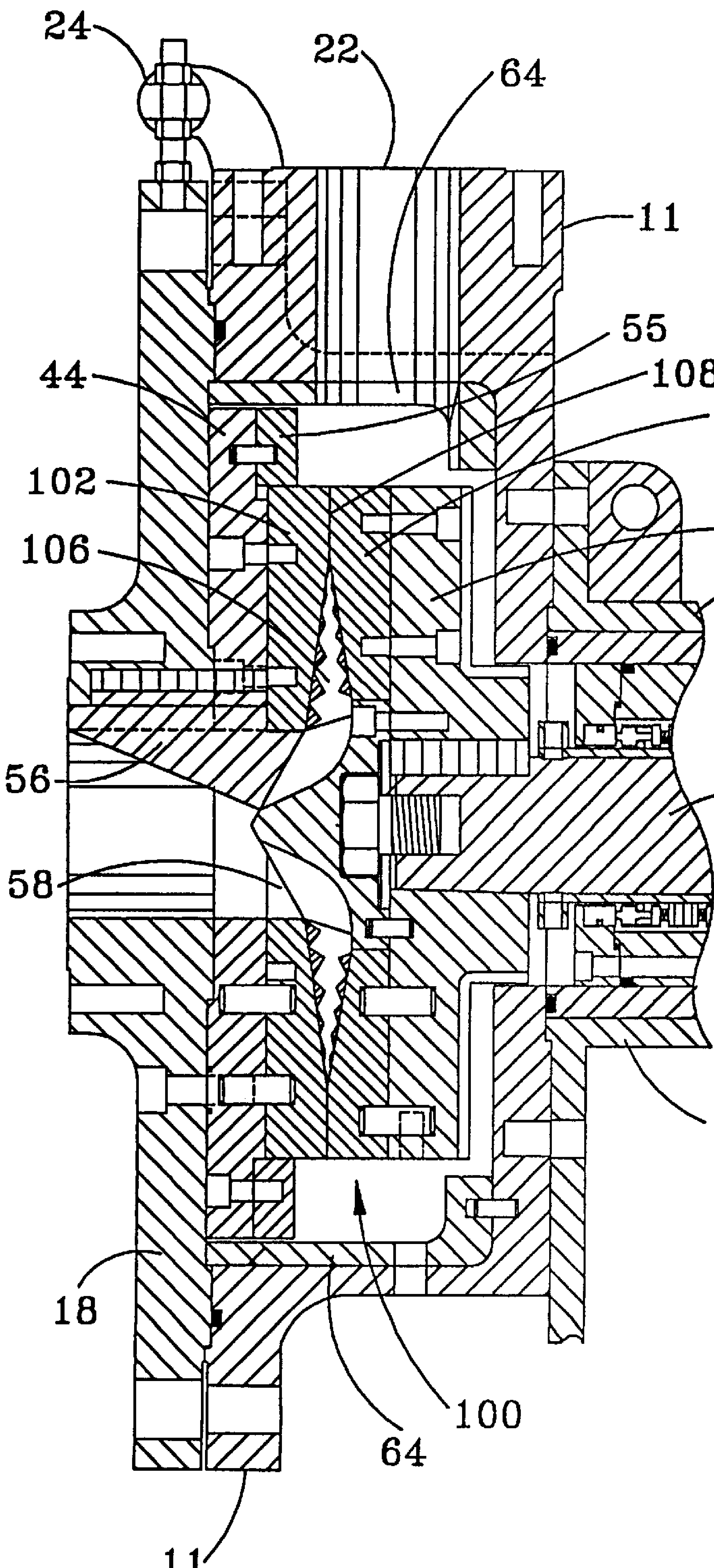

FIG. 10

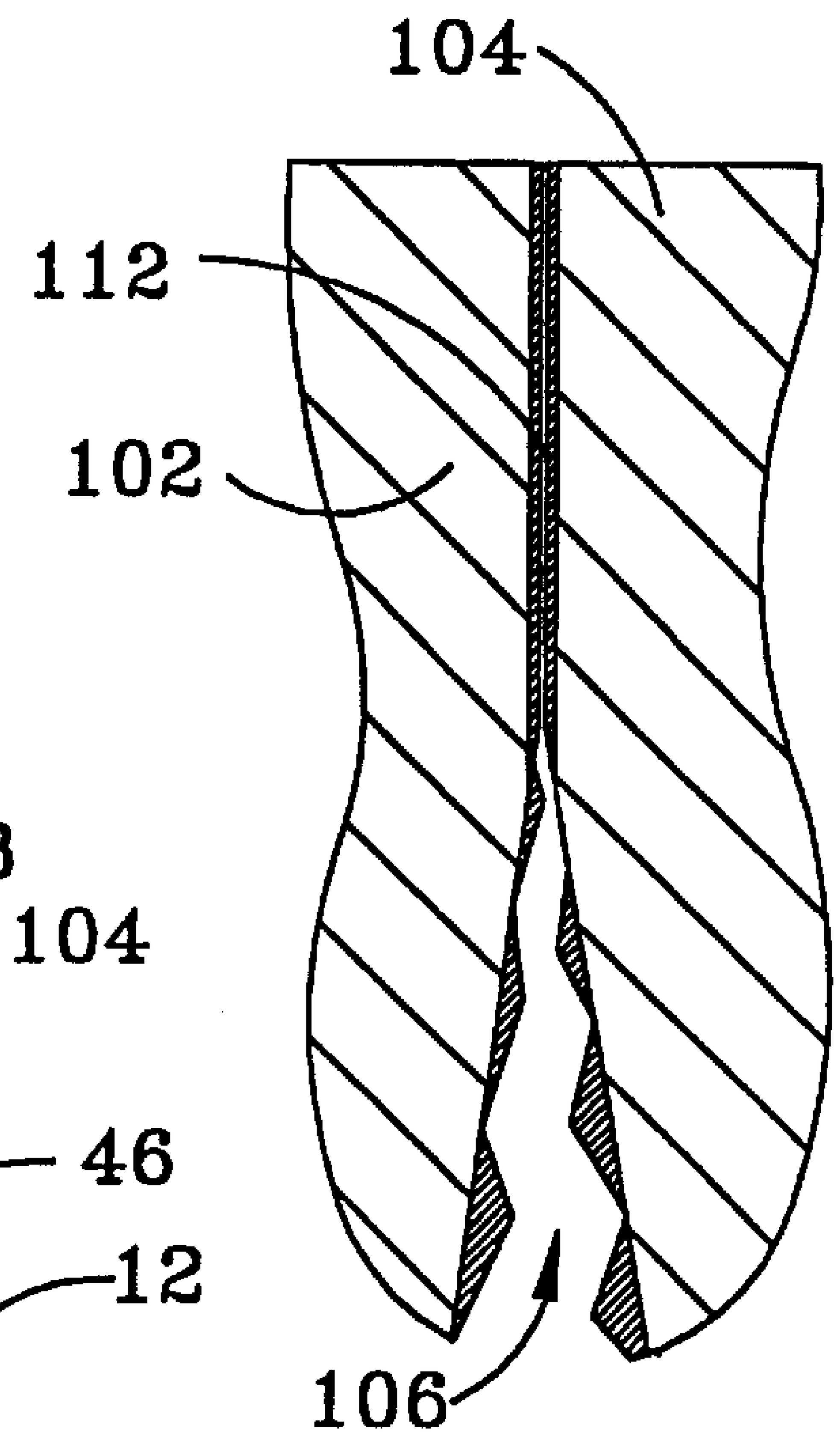

FIG. 11

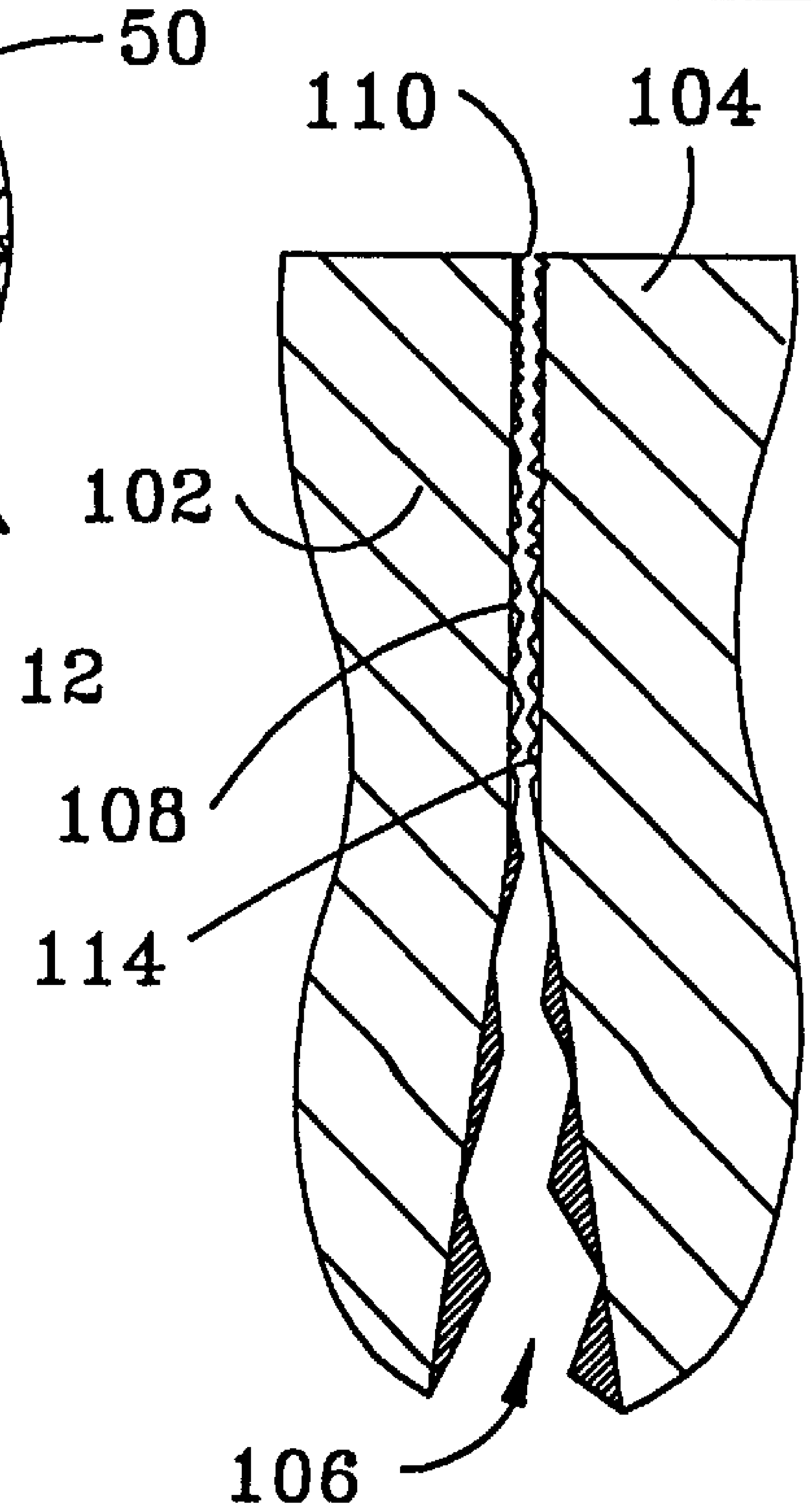

FIG. 12

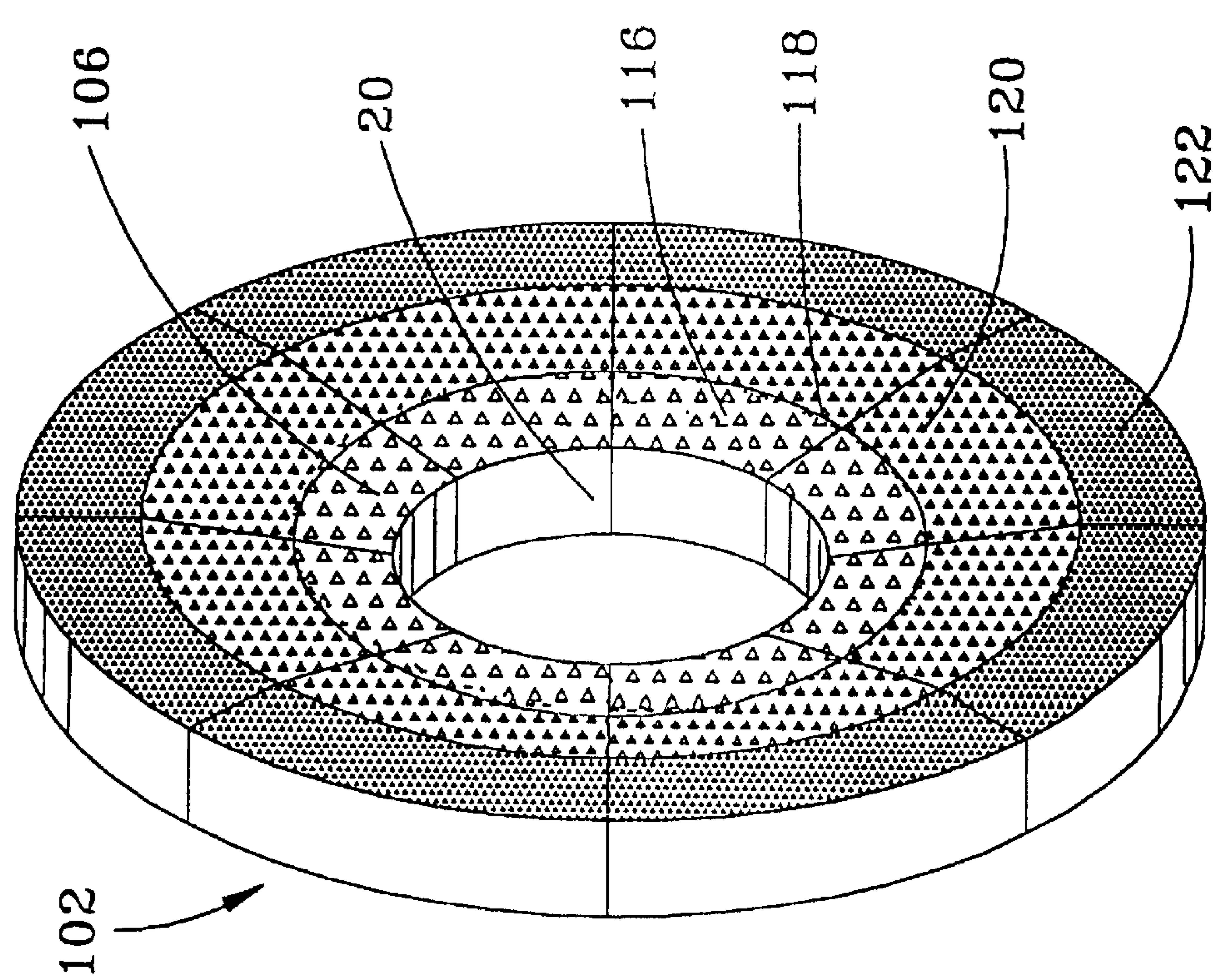

FIG. 13

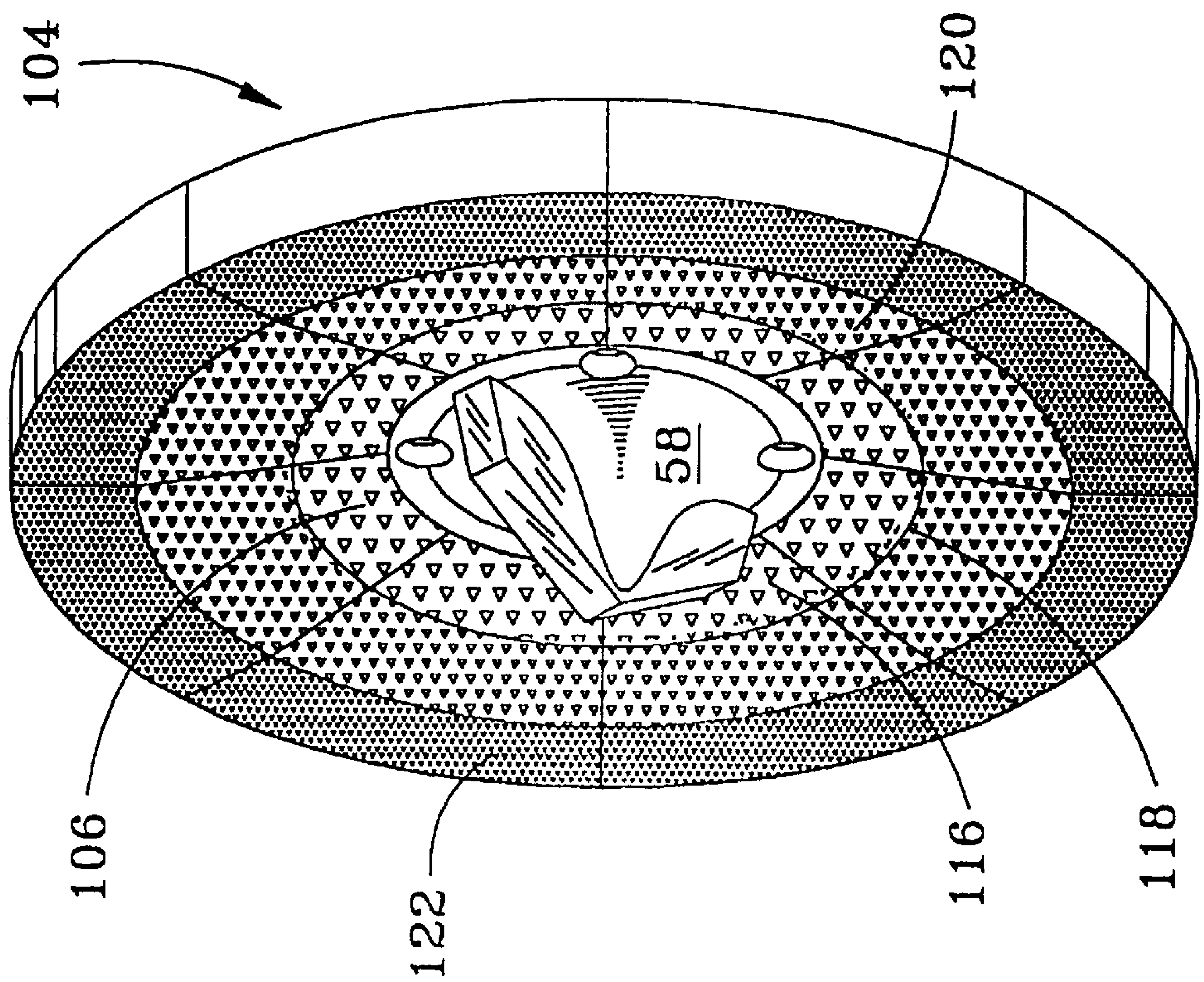

FIG. 14

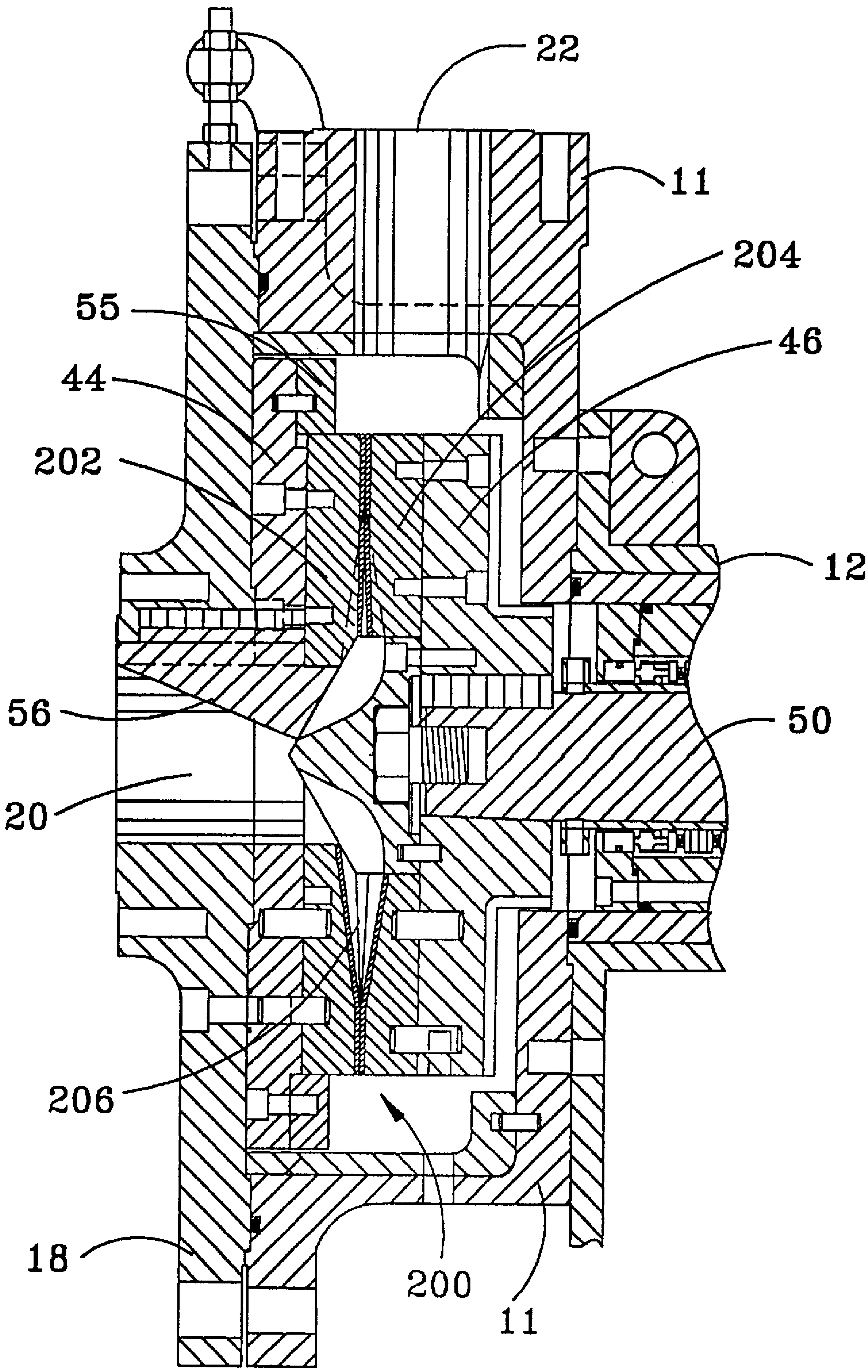

FIG. 15

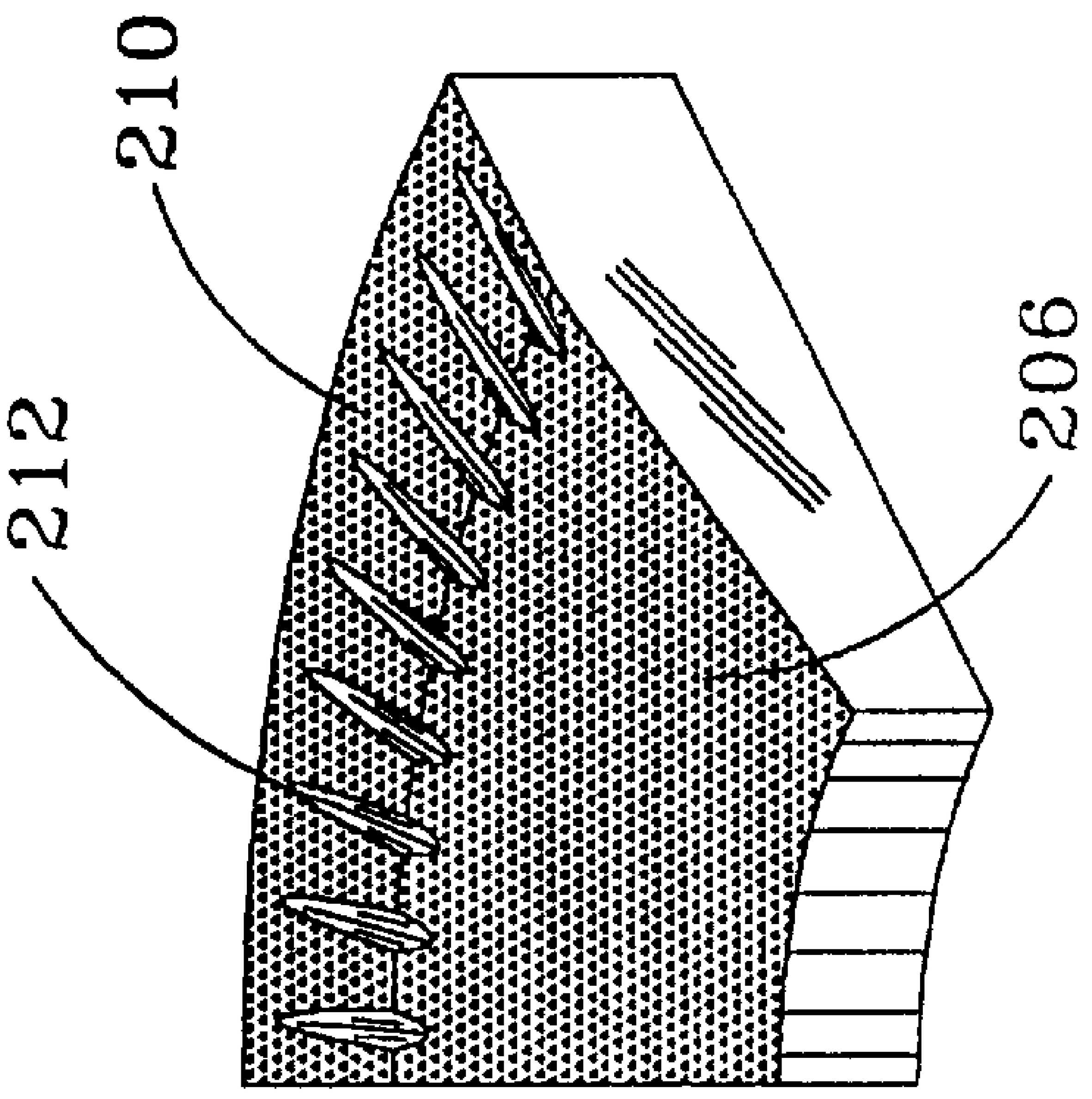

FIG. 18

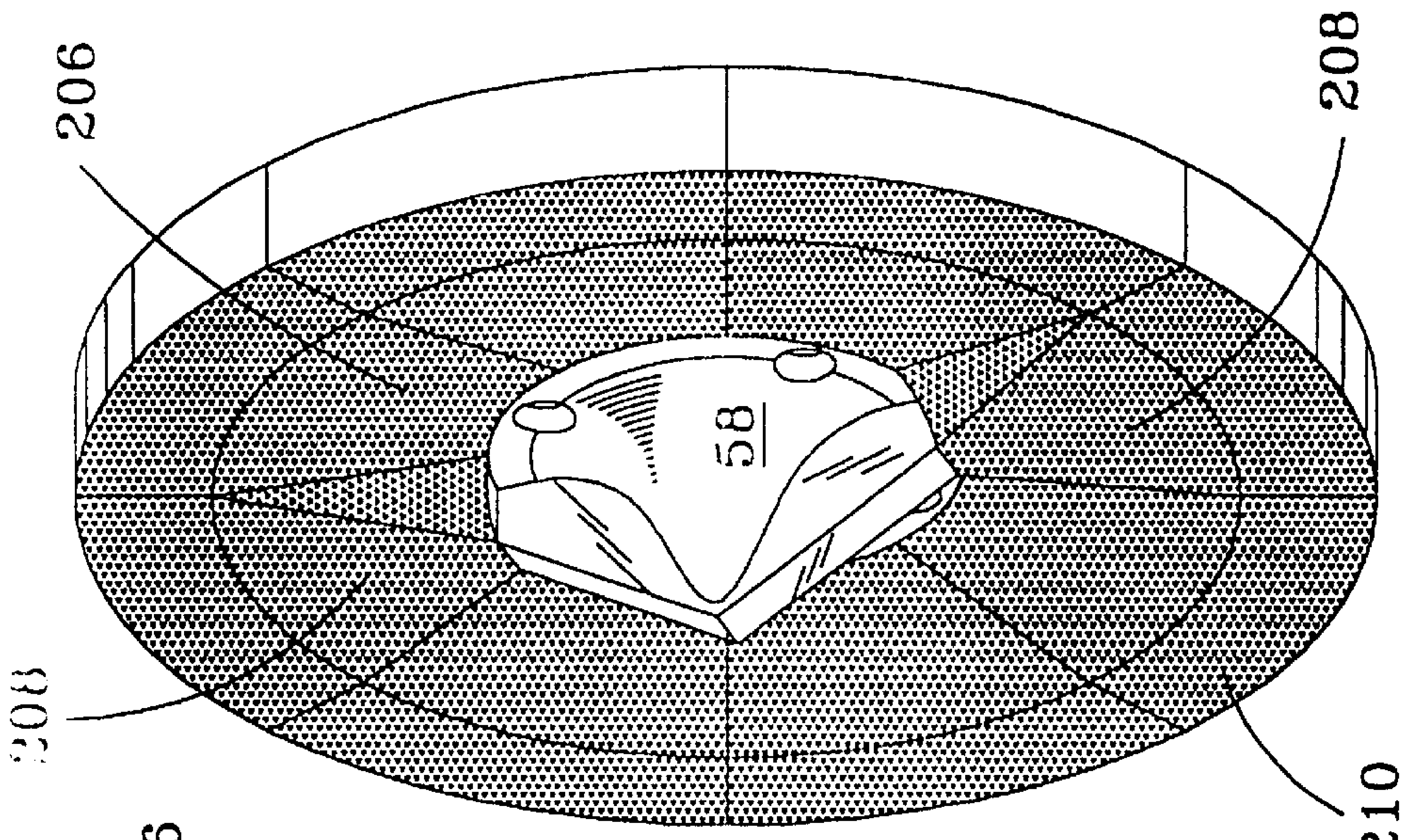

FIG. 17

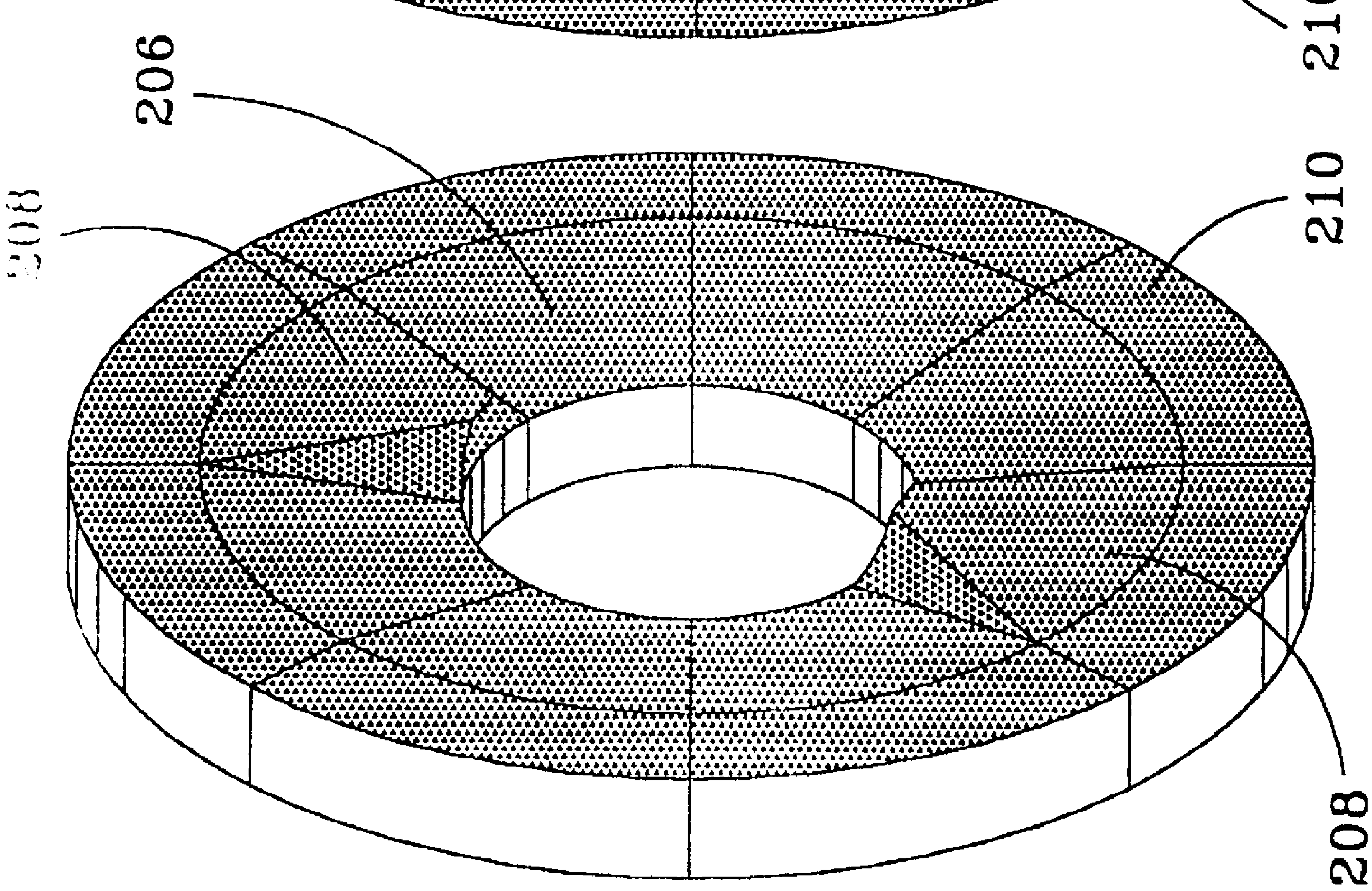

FIG. 16

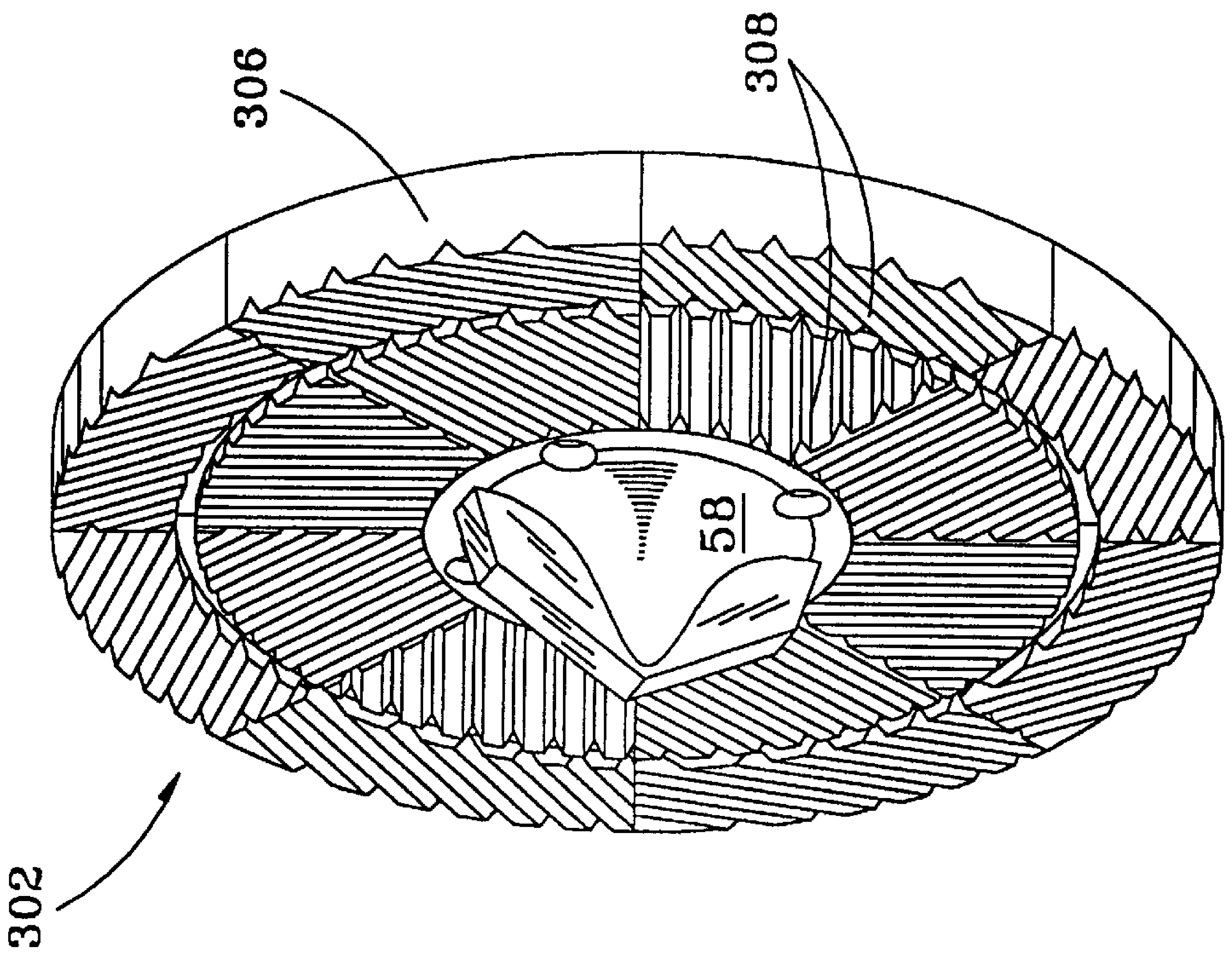

FIG. 20

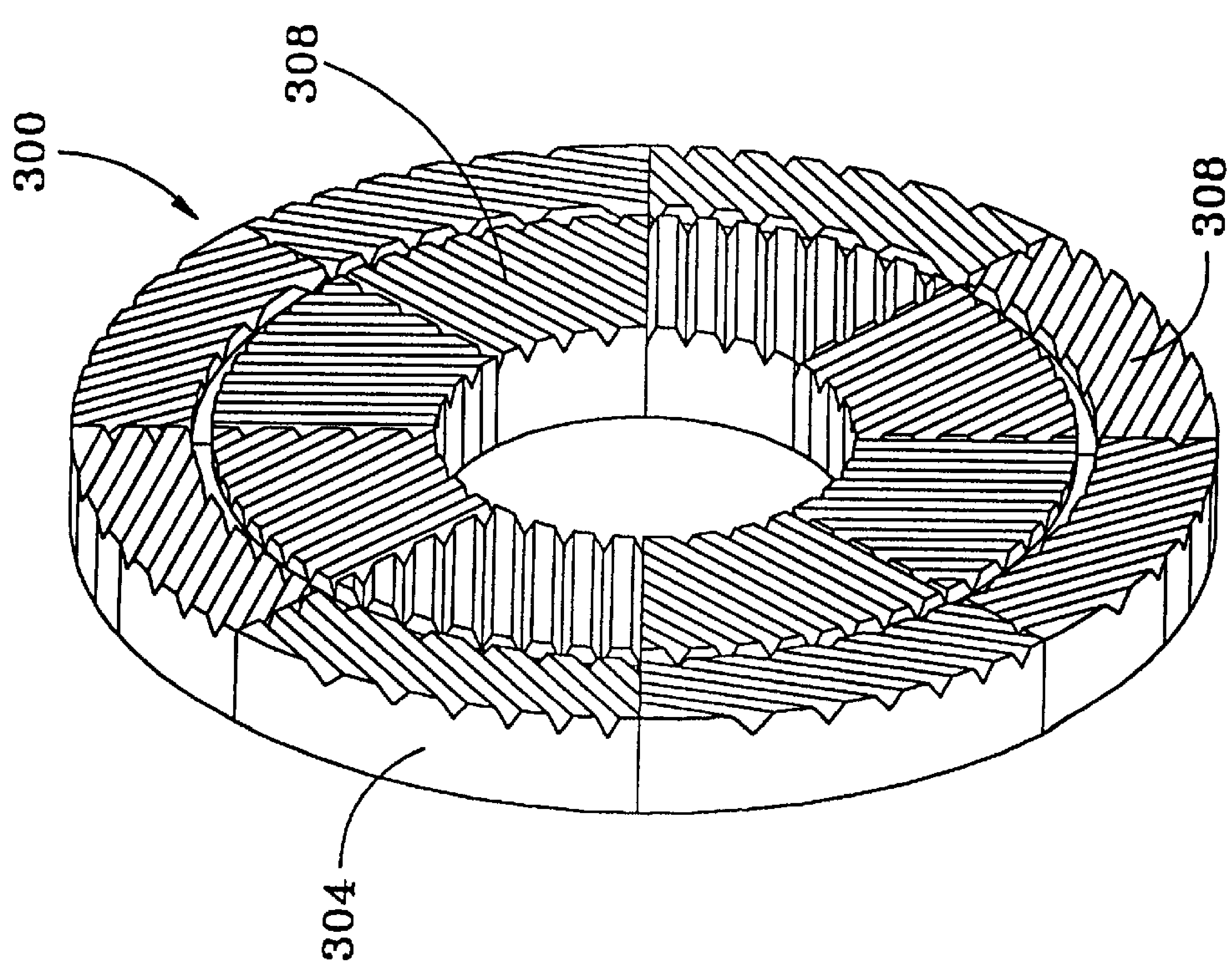

FIG. 19

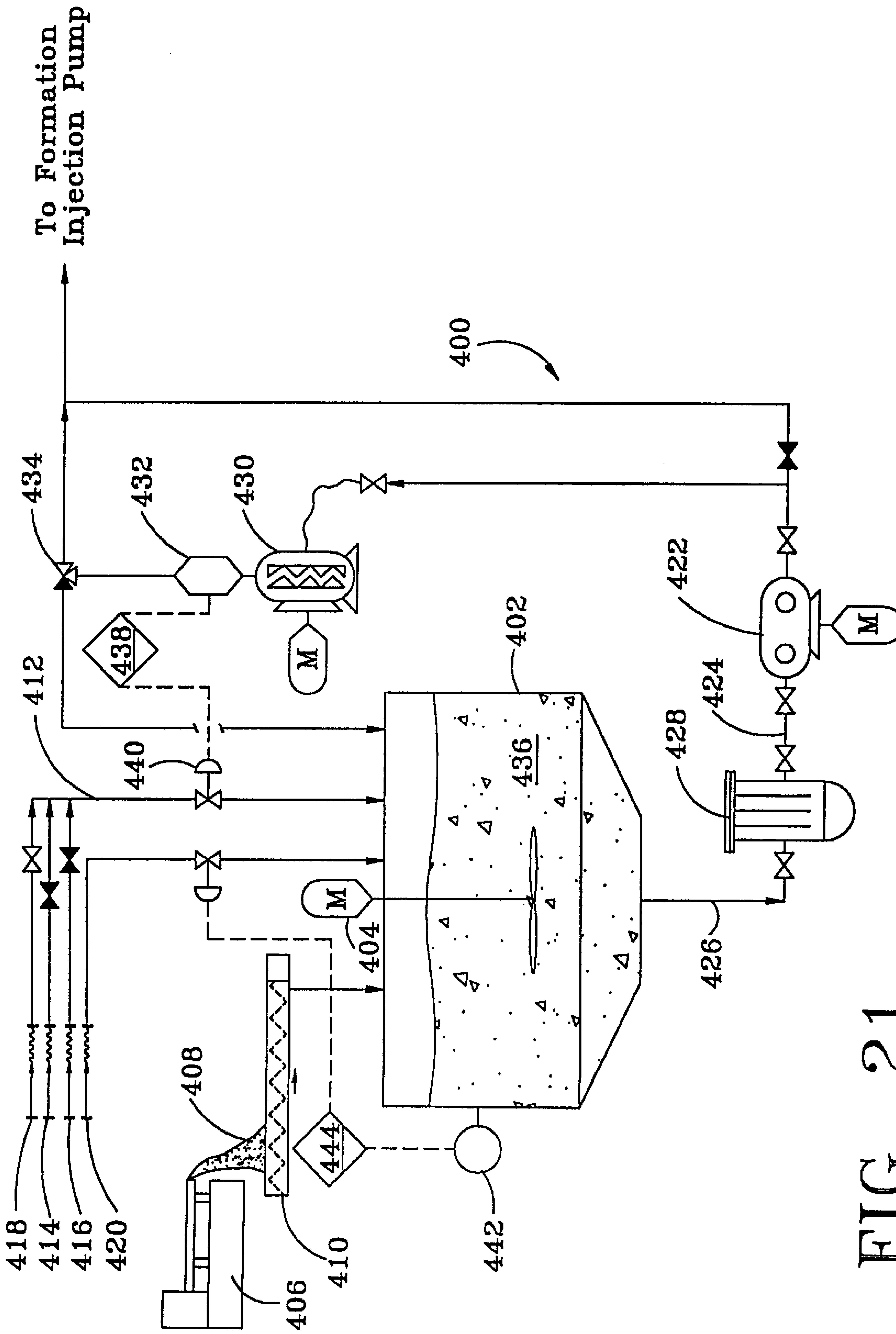
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FIG. 21

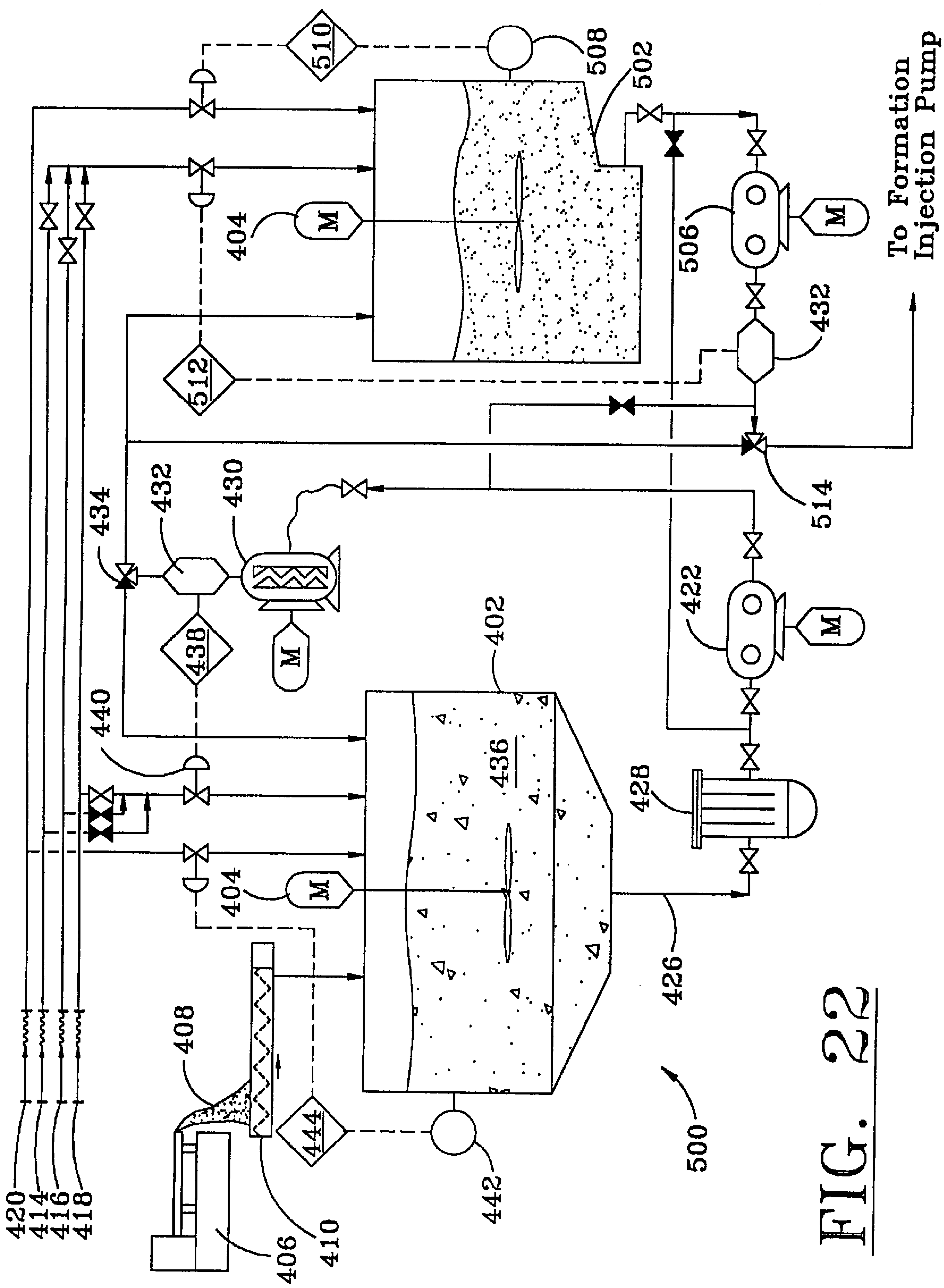
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FIG. 22

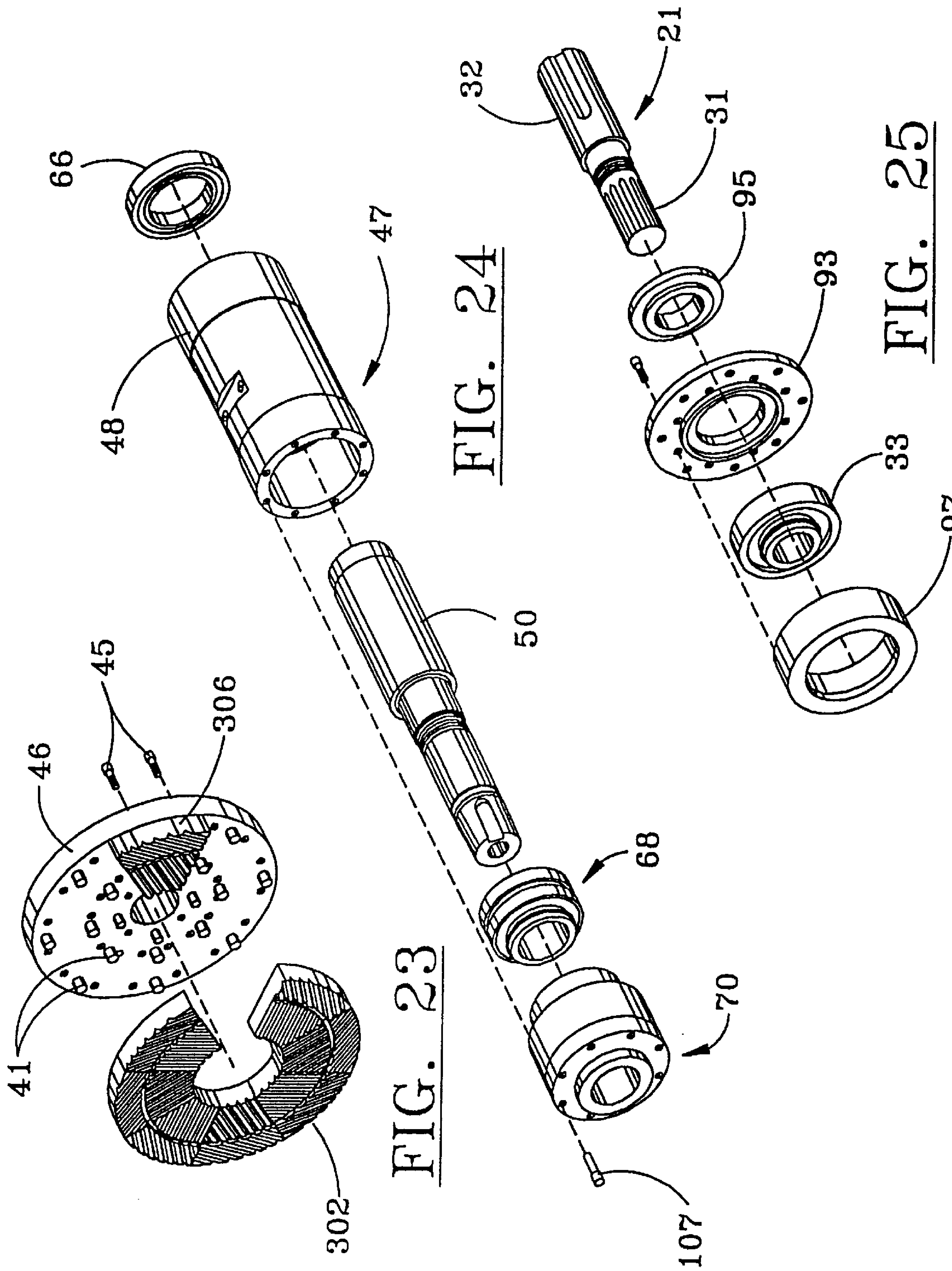

FIG. 23

FIG. 24

FIG. 25

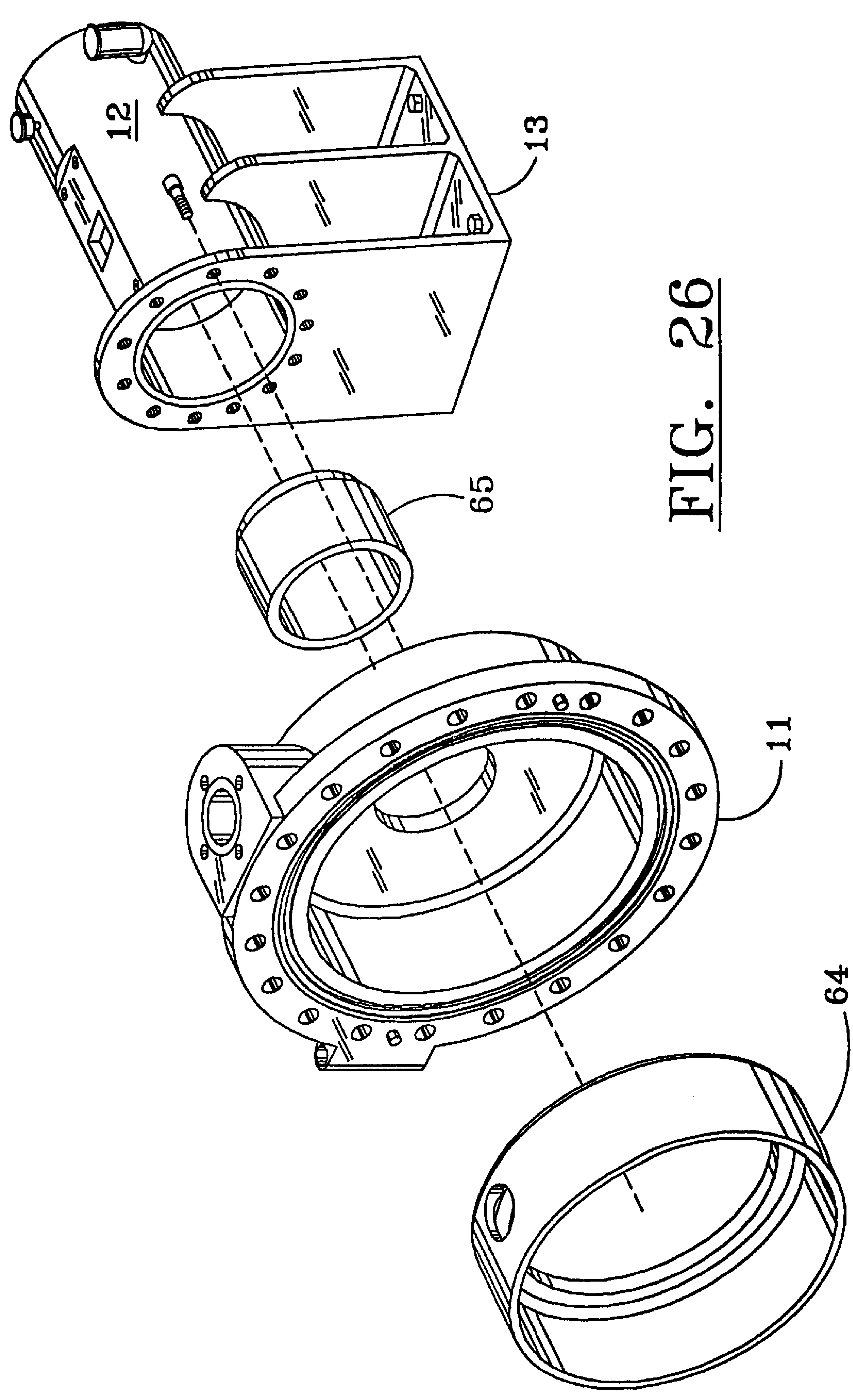

FIG. 26

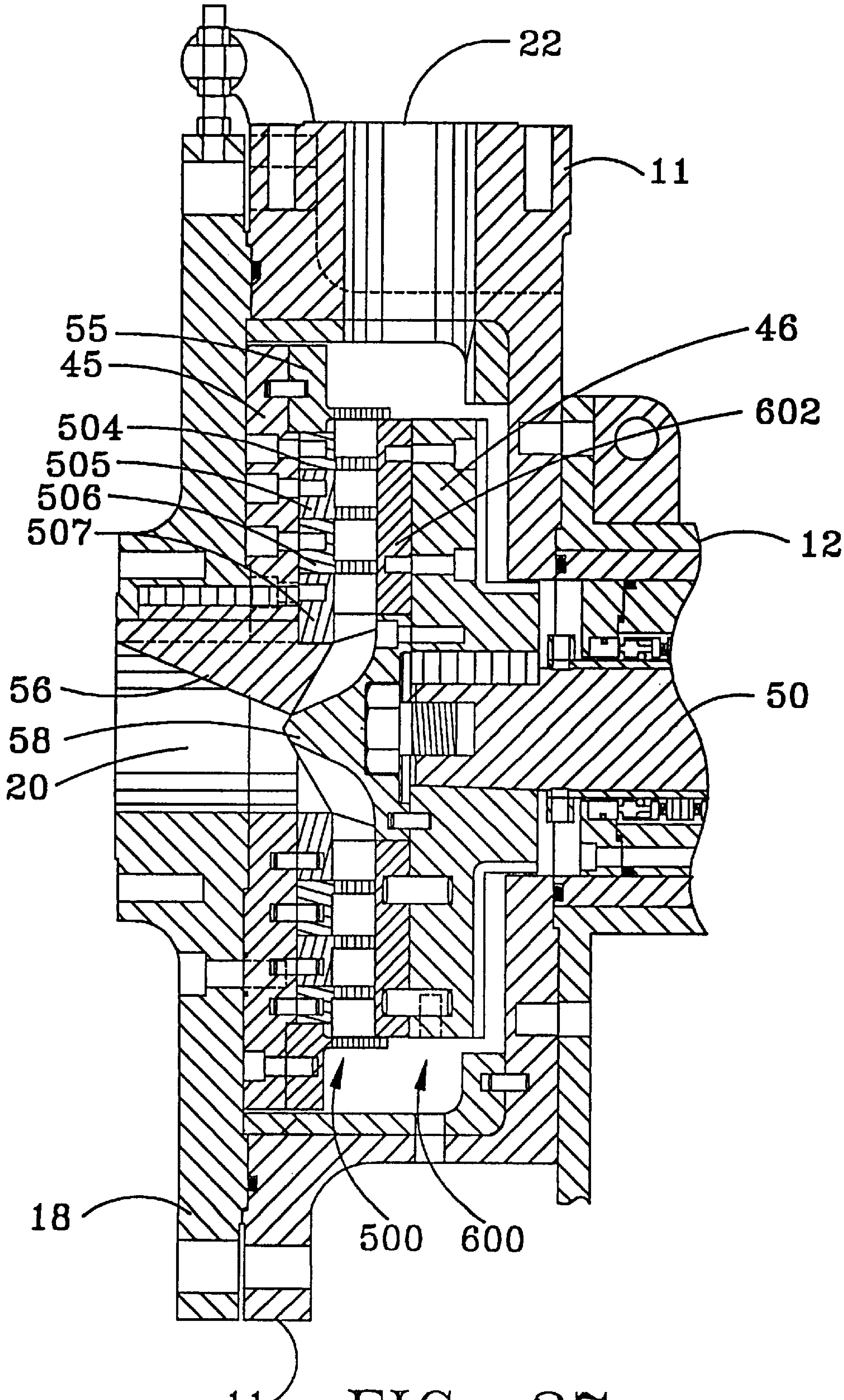

FIG. 27

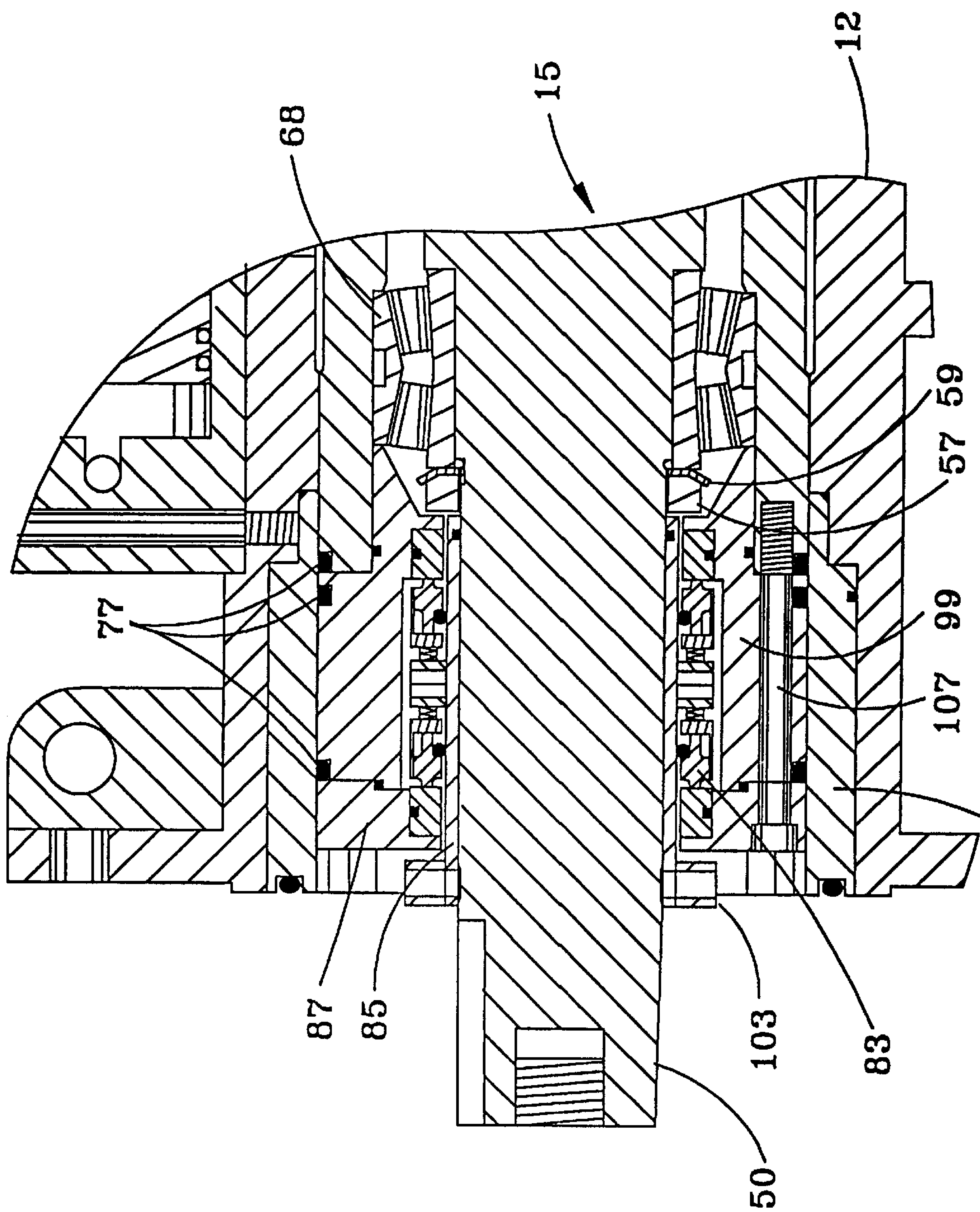

FIG. 28

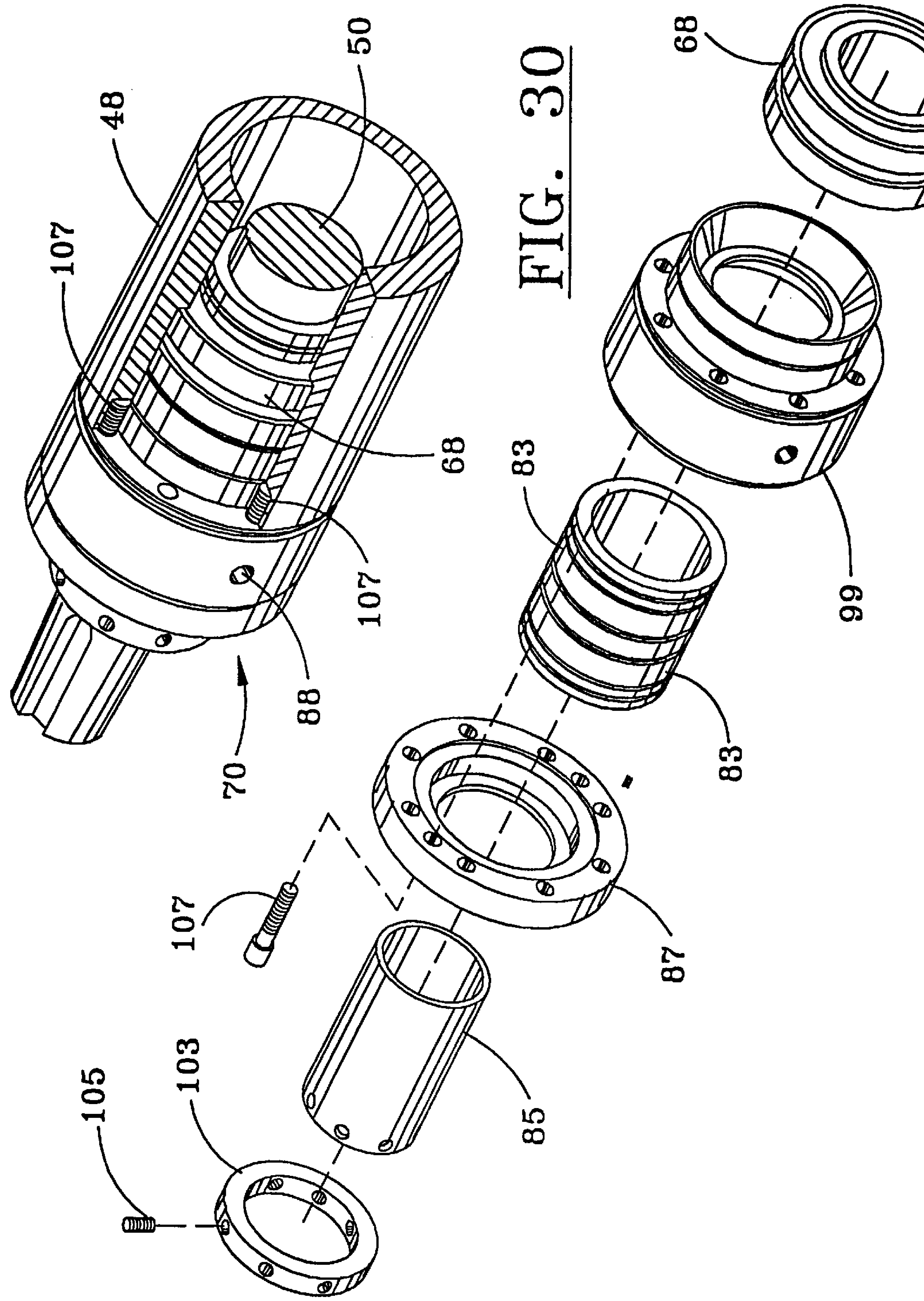

FIG. 30

FIG. 29

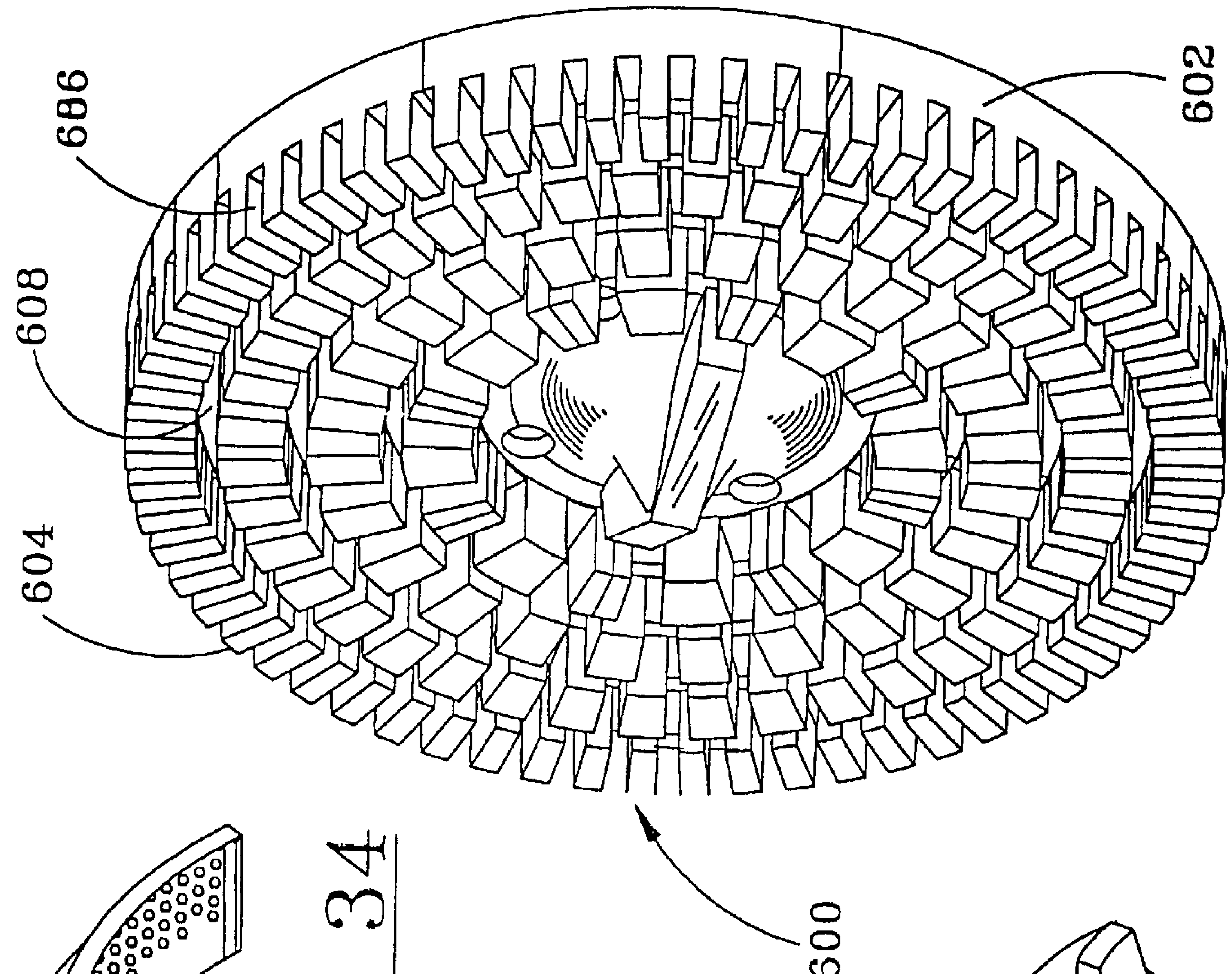

FIG. 32

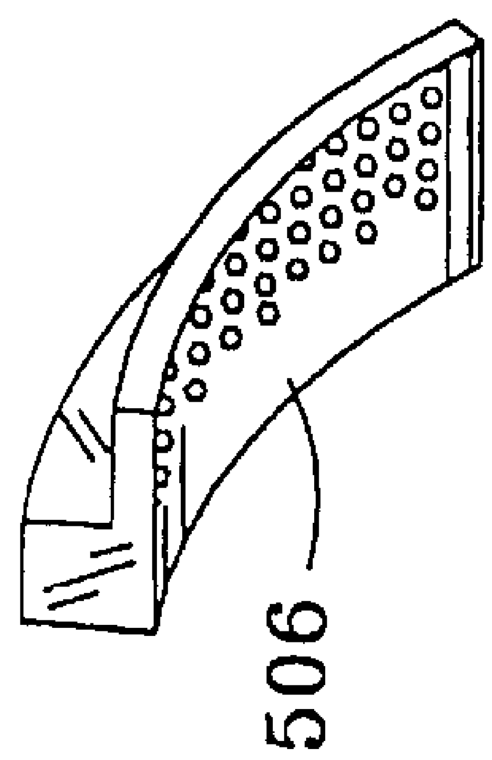

FIG. 34

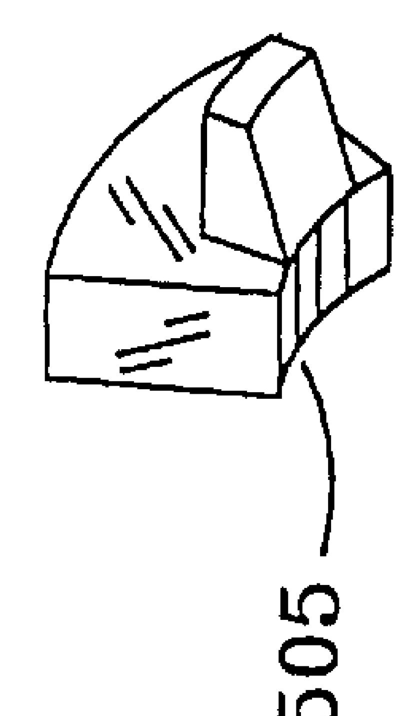

FIG. 33

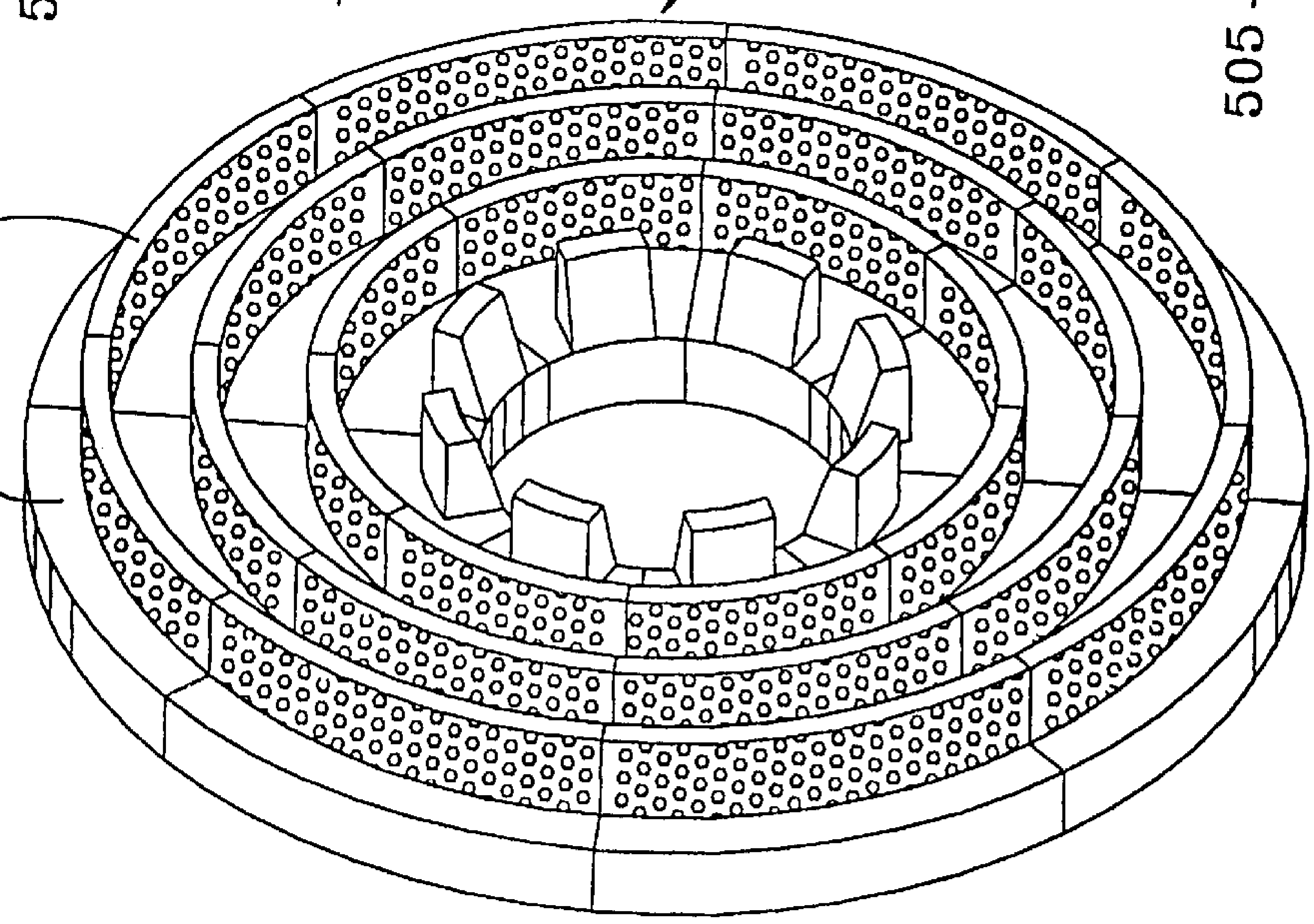

FIG. 31

ROTARY GRINDER

This application is a division of U.S. patent application Ser. No. 09/023,051, filed Feb. 13, 1998 now U.S. Pat. No. 5,971,307 which is a continuation of U.S. patent application Ser. No. 08/802,848, filed Feb. 19, 1997 which is a continuation of application Ser. No. 08/477,229 filed Jun. 7, 1995 now abandoned which is a divisional of application Ser. No. 08/368,386 filed Dec. 30, 1994 now issued as U.S. Pat. No. 5,495,986 which is a continuation of application Ser. No. 08/060,753 filed May 12, 1993 now abandoned.

BACKGROUND OF THE INVENTION

1. Field of the Invention

The present invention relates to in-line grinding or milling apparatus in general which size and disperse solids contained in a liquid slurry as they are pumped through it and more particularly to machines with adjustable rotors having a number of stator-rotor combinations interchangeably mountable in the machine to accomplish a wide variety of size reduction needs. The present invention also pertains to the use of such in-line grinding apparatus in drill cuttings disposal systems wherein the cuttings are treated and refined to form a slurry to be pumped into an earth formation through a well bore.

2. General Background

It is well known within the art that a stator-rotor assembly composed of intermeshing teeth or shear blocks may be use in the sizing of both flexible and friable solids. However, heretofore, fine grinding of such solids produced by such methods have been done by different machines, i.e. ball, or roller mills and fine shredders and the like.

An apparatus for grinding solids as they are pumped through the machine has been disclosed in U.S. Pat. Nos. 5,495,986 and 5,586,729. The apparatus disclosed the concept of utilizing an adjustable rotor in combination with intermeshing teeth or shear blocks to accomplish the size reduction of solids in a liquid slurry. An arrangement of the intermeshing teeth further discloses a tooth arrangement which allows the gap between the stator and rotor to be set for any desired particle size. However, the apparatus does not teach a structure for performing such adjustment nor does it teach a method for interchangeably adapting non-intermeshing rotor and stator elements.

It has now become evident that a need exists for fine grinding solids entrained in a slurry to micron size. Ideally such fine grinding should be accomplished with the same machine configured to receive interchangeable stator-rotor assemblies capable of shearing and or fine grinding particles to micron size. For example, the disposal of drill cuttings from drilling various types of wells has become an increasingly difficult problem due to restrictions imposed by various governmental authorities and the desire to minimize environmental damage. These problems are aggravated, or at least amplified, in certain well drilling operations, particularly in offshore drilling operations, wherein the disposal of drill cuttings normally requires transport of the cuttings to a suitable landfill or shore-based processing system.

One solution to drill cuttings disposal has been to separate the drill cuttings from the drilling fluid and reclaim coarse cuttings for use as construction grade gravel. Finer particles of material are slurried and injected into an earth formation through a disposal well. In many instances, however, disposal of all of the drill cuttings is not as conveniently handled. This is especially evident in offshore well drilling operations where the separated cuttings are not suitable for reuse, reclamation or other disposal processes. The cost of managing drill cuttings has increased dramatically as the offshore platforms migrated into deeper waters which further increases the distance to land-based disposal operations.

U.S. Pat. No. 5,129,469 illustrates a method and system for processing drill cuttings whereby drill cuttings are reduced in particle size by using a centrifugal pump as the grinding means of size reduction. After size reduction, the drill cuttings slurry is injected back into the formation through the well bore. It has been found in practice that the centrifugal pump grinding means contained in the above referenced patent has no ability to produce a consistent particle size. As a result, the system operates best when used in conjunction with a shaker screen to separate oversized solids leaving the centrifugal pump grinding means. Sized solids falling through the screen are suitable for injection, whereas rejects from the screen are recirculated repeatedly through the pump grinding means until they are sufficiently small to pass through the shale shaker screen. It is therefore evident that a more efficient method and apparatus is needed to provide a consistent particle size reduction

SUMMARY OF THE INVENTION

The present invention has been developed with a view toward providing an improved apparatus and method for interchangeably adapting an in-line disk attrition mill, having variable displacement, for use as a rotary shear and or fine particle grinder. The adaptation includes the addition of a fine grind ring surrounding an intermeshing stator-rotor assembly, thereby having the ability to yield an even finer particle size. Further adaptation includes the use of a hard surfaced stator-rotor assembly having a number of different configurations including an internal conically shaped cavity. Some stator-rotor assemblies are ground to precision tolerances. Such close running parallel perimeters allow for the production of particle sizes in the micron range. Further, the use of rotor teeth intermeshing with concentric stator rings having perforations results in high energy shearing and dispersion of solids/liquids suspensions as well as liquid/liquid suspensions, all made possible or enhanced by the ability to adjust the position of the rotor relative to the stator. Other features include the use of a mechanical seal cartridge to clamp the machine's thrust bearing in place, thus providing for a minimum of shaft deflection and run-out. This configuration improves the longevity of the mechanical seal and insures that the stator-rotor assemblies maintain alignment.

It is an object of the present invention to provide an improved apparatus for the sizing and dispersion of solids in a liquid slurry through the use of a variety of interchangeable stator and rotor assemblies.

A further object of the present invention is to provide an improved apparatus for the reduction of solids using an intermeshing configuration of stator and rotor components utilizing a stationary fine grind ring surrounding the rotating rotor in a manner whereby openings in the fine grind ring co-act with rotor teeth to dramatically increase the number of shears produced per revolution of the machine rotor.

It is also an object of the present invention to provide an improved apparatus for adjusting the position of the rotor relative to the stationary stator so that the machine can be adjusted for various drive configurations, for component wear, and for the particle size required to be produced.

An additional object of the present invention is to provide an improved apparatus for the reduction of long, stringy, or oversized solids through the use of a removable and replaceable shear bar mounted stationary in the inlet of the in-line grinder in a manner whereby the shear bar comes in close proximity to the revolving rotor hub so that material is sufficiently sheared and reduced prior to enter the grinding chamber formed between the stator and rotor.

A further object of the invention is to provide an improved method and apparatus for minimizing shaft deflection and run-out through the use of a mechanical seal cartridge to clamp the drive shaft bearing in place, thereby minimizing the overhung distance from the bearing support to the rotor.

It is a further object of the present invention to provide an interchangeable stator rotor assembly capable of the fine grinding of friable materials, such as drill cuttings, minerals, pigments, clays, and the refining of fibrous materials such as paper pulp, as well as high energy shearing and dispersion of solids and liquids.

An additional object of the present invention is to provide a system, utilizing an interchangeable adjustable stator-rotor disk attrition mill, for the refining and dispersion of drill cuttings produced during the drilling of oil and gas wells, particularly in offshore well drilling operations. In accordance with an important aspect of the present invention, drilling cuttings returned to the surface are separated from the drilling fluid, mixed with a suitable liquid, such as sea water, and circulated and sheared by an in-line grinder to reduce the cuttings particles to a size which forms a slurry-like composition which may be pumped into a selected zone in a wellbore for disposal.

In accordance with yet a further aspect of the present invention, there is provided a system which is advantageously used in conjunction with offshore well operations for receiving drill cuttings, reducing the size of the drill cuttings, and blending the drill cuttings with a suitable carrier liquid, such as sea water or waste water, so that a slurry-like composition may be pumped into a wellbore, preferably into and through an annular zone between the wellbore casing and an earth formation, for fracturing and permeation into the formation. The system is particularly effective, compact and adapted for use in conjunction with offshore well drilling operations.

In accordance with another aspect of the present invention, a method is presented for automatic control of the drill cuttings preparation process whereby the density, viscosity, and flow characteristics of the injected cuttings can be automatically monitored and adjusted to provide optimum conditions to promote permeation of the formation and to prevent its plugging.

Those skilled in the art will recognize the above-described advantages and superior features of the invention together with other aspects thereof upon reading the detailed description which follows in conjunction with the drawings.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. **1** is an isometric view of the preferred grinder embodiment;

FIG. **2** is a plan view of the preferred grinder embodiment shown in FIG. **1**;

FIG. **3** is a cross-section view of the preferred grinder embodiment taken along sight line **3**—**3** shown in FIG. **2**;

FIG. **4** is a partial cross-section illustrating a second embodiment of the rotor adjustment mechanism shown in FIG. **3**;

FIG. **5** is an isometric view of the prior art segmented stator assembly shown in cross section in FIG. **3**;

FIG. **6** is an isometric view of the prior art segmented rotor and hub assembly shown in cross section in FIG. **3**;

FIG. **7** is an isometric view of the fine grind ring shown in cross section in FIG. **3**;

FIG. **8** is a partial cut-a-way isometric view of the casing cover, stator plate and segmented stator illustrating breaker bar location;

FIG. **9** is a partial exploded isometric view of the principle elements of the preferred embodiment;

FIG. **9**A is a continuation of the exploded view of the principle elements of the preferred embodiment illustrated in FIG. **9**;

FIG. **10** is a partial cross section view of the preferred grinder embodiment with non-intermeshed stator-rotor assembly;

FIG. **11** is an enlarged partial cross-section view of the with non-intermeshing stator/rotor assembly illustrated in FIG. **10**;

FIG. **12** is an enlarged partial cross-section second embodiment of the stator/rotor assembly;

FIG. **13** is an isometric view of the segmented non-intermeshing stator assembly;

FIG. **14** is an isometric view of the segmented non-intermeshing rotor and hub assembly;

FIG. **15** is a partial cross-section view of the preferred grinder embodiment with a third embodiment for the non-intermeshing stator-rotor assembly;

FIG. **16** is an isometric view of the third embodiment of the segmented non-intermeshing stator illustrated in cross-section in FIG. **15**;

FIG. **17** is an isometric view of the third embodiment of the segmented non-intermeshing rotor illustrated in cross-section in FIG. **15**;

FIG. **18** is a partial isometric view of a fourth embodiment of the segmented non-intermeshing rotor and stator illustrated in FIGS. **16** and **17**;

FIG. **19** is an isometric view of a fifth embodiment of the segmented non-intermeshing stator;

FIG. **20** is an isometric view of a fifth embodiment of the segmented non-intermeshing rotor;

FIG. **21** is a schematic diagram illustrating a process utilizing the preferred grinder embodiment for processing drill cuttings;

FIG. **22** is a schematic diagram illustrating a second process utilizing the preferred grinder embodiment for processing drill cuttings;

FIG. **23** is an exploded view of the rotor assembly and rotor plate illustrated in FIG. **9**A;

FIG. **24** is an exploded view of the rotor shaft and quill assembly illustrated in FIG. **9**A;

FIG. **25** is an exploded view of the drive assembly illustrated in FIG. **9**A;

FIG. **26** is an exploded view of the case assembly Illustrated in FIG. **9**A;

FIG. **27** is a partial cross section view of the preferred grinder embodiment with combination intermeshed stator shear ring assembly and segmented rotor assembly having teeth similar to that shown in FIG. **6**;

FIG. **28**. is an enlarged view of a portion of the cross section view in FIG. **3**;

FIG. **29** is an exploded view of the mechanical seal and bearing assembly illustrated in FIG. **24**;

FIG. **30** is a partial cut-a-way cross section isometric view of the mechanical seal and bearing assembly shown in FIG. **29** as assembled on the rotor shaft;

FIG. 31 is a segmented stator shear ring assembly;

FIG. 32 is a segmented rotor having teeth cooperative with teeth and rings of FIG. 31;

FIG. 33 is a detailed isometric view of a tooth segment illustrated in FIG. 31; and

FIG. 34 is a detailed isometric view of a ring segment illustrated in FIG. 31.

DESCRIPTION OF PREFERRED EMBODIMENTS

In the description which follows, like parts are marked throughout the specification and drawing with the same reference numerals, respectively. The drawing figures are not necessarily to scale and certain features may be shown in schematic form in the interest of clarity and conciseness.

The in-line grinder **10** shown in FIG. **1** includes a casing **12** and a grinding chamber housing **11** better seen in FIG. **2** which houses the segmented stator and rotor assemblies **14,16** shown in FIGS. **5** and **6** and disclosed in the prior art or variations thereof illustrated in FIGS. **13–20** which collectively accomplish product size reduction as required. The casing **12** is generally supported by a mounting frame member or base portion **13**. The stator/rotor assemblies are accessed through the removal of the casing cover **18** via bolts **17**, which is supported by a pivot arm **24**. The position of the rotor assembly **14** relative to the stator assembly **16** is adjusted through the use of a hydraulic actuator mechanism **26** located at the top of the casing **12**. The hydraulic actuator **26** is also supplied with a pressure relief valve **28** on the opening side of the cylinder **26** so that the rotor assembly **16** can instantaneously move away from the stator assembly **14** in the event that the grinder **10** ingests a solid object of a size larger than the presetting and which it cannot shear. The pressure relief valve **28** can be adjusted to relieve at any setting desired as a means of tailoring the machine to individual applications. Further, the minimum gap and maximum gap which can be achieved between the stator assembly **14** and rotor assembly **16** can be manually set through adjustment of the stop gap set screws **30** shown adjacent the hydraulic actuator **26**.

In operation, coarse solids slurried in liquid are introduced by pump into the suction port **20** located at the center of the casing cover **18**. Sized solids are discharged through the outlet port **22** located at the top of the grinding chamber housing **11** adjacent the stator/rotor assemblies **14, 16** and at a right angle to the suction port **20**. The rotor assembly **16** is rotated relative the stationary stator assembly **14** via an input shaft **32** which extends beyond the casing **12** and driven by a prime mover such as a motor. Casing vent **32**, grease zerk **36** and oil cup **38** are also provided for ventilation and lubrication of the sliding and rotating components within the case **12**.

A cross-section of the preferred in-line grinder embodiment is shown in FIG. **3**. This cross-section is taken along the center line of the machine shown in FIG. **2** and features the intermeshing configuration of stator and rotor assemblies **14,16**. FIG. **3** illustrates several unique features of the in-line grinder which allow the apparatus to be configured for individual size reduction applications. These features include: interchangeable stator and rotor segments **40, 42** seen in FIGS. **5** and **6** which are mounted on corresponding stator and rotor plates **44,46**, such stator and rotor segments being also interchangeable with segments having various face configurations; an adjustable quill assembly comprising the quill **48**, a quill shaft **50** and a unique mechanical seal and bearing arrangement **68,70** design to minimize shaft deflection and run-out, and a quill arm **52** which allows the gap between the stator and rotor assemblies **14,16** to be set for individual applications; an optional fine grind ring **54** surrounding the stator and rotor assemblies **14,16** to increase the number of shears taking place in the grinder; and a removable breaker bar **56** located in the suction port of the grinder **10** to reduce inlet solids to a selected size prior to grinding.

As best seen in FIG. **9**, stator assembly **300** and its segments **304** are mountable to a single stator plate **44** which is in turn mounted to the casing cover **18**. Likewise, rotor assembly **302** and its segments **304** are mountable to the rotor plate **46** seen in FIG. 9A, which is also keyed via a shaft key **35** and retained to the quill shaft **50** via a rotor bolt **60** and washer **61**. The rotor hub **58** seen in FIG. 9A covers the rotor mounting bolt **60** and directs flow into the grinding chamber formed by the co-action between the stator and rotor assemblies **300, 302**. The rotor hub **58** may also be supplied with one or more flights **62**, illustrated in FIG. **6**. This arrangement allows the mounting of a multitude of different configurations of stator and rotor segments, seen in FIGS. **13–20**, within the same grinder chamber **9** configuration seen in FIG. **3** in order to tailor the grinder **10** to individual applications. These stator/rotor assemblies may have intermeshing features as seen in FIGS. **5** and **6** or non-intermeshing features as seen in FIGS. **13–20**, depending upon the application. However, FIG. **3** illustrates rotor and stator grinding assemblies **14,16** as being the intermeshing type detailed in FIGS. **5** and **6** and discussed in our previous patents, as contained within the grinding housing **11** attached to the front portion of the casing **12** and accessed by opening the casing cover **18**. As also seen in FIG. **3** the interior grinding chamber **9** of the grinding housing **11** is protected from wear by a replaceable wear ring **64** as best seen in FIG. **26**.

As seen in FIG. 9A the quill assembly **47** is held in a slidable position inside the casing assembly **15** and further retained in linear non-rotatable relationship with the case portion **12** by the quill arm **52** attached to the quill assembly **47** and passing through a slot **19** in the case portion **12**. The drive assembly **21** attaches to the end of the casing **12** and is spline linked to the quill assembly **47**, thereby allowing linear travel of the quill assembly **47** relative to the drive assembly **21**. The hydraulic actuator assembly **26** is positioned over the quill arm and attached to the case portion **12**, thus providing remote sensing or control of the quill assembly **47** thereby effecting positioning of the stator/rotor spacing.

As better seen in FIG. **24**, the quill assembly **47** is comprised of a quill or rotor shaft **50** slidable within the quill body **48** rotatable within a unique mechanical seal **70** and thrust bearing assembly **68** integral with and attached to the front portion of the quill **48**. The thrust bearing **68** is held in position by its inner race located on a shoulder of the shaft **50** and secured by a threaded lock nut **57** and washer **59** with its outer race in contact with an inside bore of the quill **48**. This arrangement is essential in minimizing shaft deflection and run-out. The arrangement further improves the life of the mechanical seal **70** and maintains critical alignment between intermeshing stator and rotor assemblies **14,16** seen in FIG. **3** and detailed in FIGS. **5** and **6**. A roller bearing **66** is also secured to the rear portion of the shaft **50** for supporting the shaft within the quill **48**.

Since the rotor **46** and quill assembly **47** all move linearly as a single unit inside the bore of the casing **12**. a wear sleeve **65** is provided as seen in FIG. **26**. Seal rings **77** are also provided in grooves around the exterior of the mechanical seal assembly **70** and at each end of the quill **48** in slidable contact with the wear sleeve **65** and interior bore of the quill **48**. An internal spline in the rear end of rotor or quill shaft **50** is cooperative with external spine on the front end of the drive shaft **32**, thereby making a slidable connection **31** between the quill shaft **50** and the drive assembly **21**, as seen in FIG. **3** in a manner so that the quill assembly **47** can be moved linearly while the drive shaft **32** remains fixed. As seen in FIG. **25** the drive assembly **21** is comprised of a stub shaft **32** which has an external spline **31** at one end and an exterior portion **91** which may be splined or keyed as required at the opposite end; a bearing flange **93** having a lip seal **95** attached to one side and a bearing housing **97** secured to the opposite side, the bearing flange **93** which is fixable to the end of the casing **12**; and a bearing **33** rotatable about the stub shaft **32**, the bearing and bearing housing is located inside the longitudinal bore of the casing **12**.

As seen in FIG. **3**, the quill assembly **47** held in position by the quill arm **52** which projects through the casing **12** and attached to the sliding quill **48** is in turn held in position by the piston **72** contained inside the hydraulic actuator **26**. With this arrangement, application of pressurized hydraulic fluid to either side of the piston **72** will cause the entire rotor **46** and quill assembly **47** to move in unison. The range of possible movement of the quill arm **52** can be adjusted through the use of the stop gap set screws **74** located on top of the hydraulic actuator **26**. The relief valve **28** seen in FIG. **1** may be adjusted to relieve at any pressure desired by referencing the pressure gauge **76** mounted on the end of the hydraulic actuator assembly **71**.

An optional fine grind ring **54**, seen in FIG. **7**, may be mounted to the stationary stator plate **44**, thereby surrounding the rotating rotor assembly **16** with a minimum of clearance between the two. The fine grind ring **54** may be furnished with a variety of hole sizes in the ring portion of the piece so that a desired particle size can be selectively produced. Each hole **53** in the fine grind ring co-acts with each tooth **43** on the outer stage of the rotor **16** seen in FIG. **6** to drastically increase the number of shears occurring in the grinder. For example, a machine turning 1800 RPM will produce approximately 15 million shears per minute without a fine grind ring **54**. The addition of a fine grind ring **54** having ¼" diameter holes can increase the number of shears occurring to 62 million per minute. A ring with one eighth inch diameter holes can be made to produce 212 million shears per minute and 1⁄16" diameter holes to produce 800 million shears per minute. The ring portion of the piece is typically perforated with holes to yield a pattern which is 40% open. The unique use of a fine grind ring with ¼" diameter holes in association with the intermeshing stator and rotor shown in FIGS. **5** and **6** has proven to reduce 80% of a limestone gravel sample to 178 microns and 10% of the sample to 140 microns. An even finer particle size is possible through the use of a fine grind ring having smaller perforations.

As seen in FIGS. **8** and **9**, a removable breaker bar **56** may be inserted into a retainer rod **82** and inserted corresponding into a cavity **84** in the casing cover **18** and stator plate **44** to reduce oversized materials which are too large or stringy to be ingested into the suction port **20** of the grinding chamber formed by the stator and rotor assemblies **14,16**. The stationary breaker bar **56** co-acts with the flighting **62** on the rotor hub **58** as shown in FIG. **3** to shear material and provide the first size reduction stage. For applications not needing this feature, a cavity insert blank **86** may be used which fills and protects the breaker bar cavity **84** formed in the casing cover **18**. The breaker bar **56** is typically heat treated to a Rockwell "C" scale hardness of 65, but it can also be overlaid with tungsten carbide or diamond chips.

FIG. **4** illustrates a manual means of adjusting the rotor-quill assembly **47** through the use of a gap adjustment shaft **90**. The gap adjustment shaft **90** is captured between two stationary blocks **92** mounted to the casing **12** and the center portion of the gap adjustment shaft **90** is threaded. The quill arm **52** is likewise threaded so that rotating the shaft **90** causes the quill arm **52** to move. A lock nut **96** is used to clamp the gap adjust shaft **90** in place after adjustments have been made. Gap stop set screws **94** are likewise used to govern the extreme travel of the quill arm **52** in either direction.

FIGS. **5** and **6** illustrate one of the many intermeshed stator-rotor configurations mountable within the in-line grinder. A cross-section of this configuration is included in FIG. **3**. The teeth protruding from the surface of the stator and rotor travel in the valleys formed in the opposing segments so that the teeth actually intermesh and co-act with one another to shear material as it travels radially from the suction **20** to the discharge port **22** of the grinding apparatus **10**. It is essential that the stator and rotor assemblies **14,16** be segmented to allow heat treating of the components to a Rockwell "C" scale of 65 without distortion. Heat treating a single piece stator and rotor produces unacceptable distortion and thus renders them unusable.

A similar combination of intermeshing stator and rotor assemblies is shown in cross section FIG. **27**, and in detail in FIGS. **31** and **32**. The stator assembly **500** shown in FIG. **31** is composed of a series of concentric rings **502** which have holes or perforations in them. The holes in the stator rings can be ¼", ⅛" or 1⁄16" in diameter and the open area of the perforations is typically 40% of the ring's surface area. The rings **502** and their base portion **504** may be made as a single diametrical element or in pie shaped segments as seen in FIG. **34** forming a diametrical disk. The rings **502** and their base portion **504** may be provided as one piece diametrical rings forming concentric circles and attached to the stator base plate **44** via dowels **41** as illustrated in FIG. **27**. The rings **502** and base portion **504** as seen in FIG. **34** may also be furnished as segmented rings, necessary only if they are to be heat treated to improve wear resistance, in which case a series of pie shaped concentric ring segments as seen in FIG. **34** forms a wedge or pie shaped segment for attachment to the stator plate **44** shown in FIG. **27**.

A row of teeth segments **505** as seen in FIG. **31** may also be integrated into the pie shaped segments shown in FIG. **31** or as a one piece diametrical concentric ring and also attached to the stator base plate **44**. These teeth segments **505** are cooperative with the rotor tooth segments **604** illustrated in FIG. **32**. The rotor illustrated in FIG. **32** may also be made in one piece **33** in diametrically concentric rings or in pie shaped segments which bolt to the rotor base plate **46** in identically the same manner as the rotor assemblies described earlier. The segments **602** may be provided as investment castings with rows of teeth **604** having gaps **606** between teeth and gaps **608** between rows which get progressively narrower and more numerous in each successive row emanating from the epicenter of the disk. The sides of the teeth **604** are perpendicular to their base **602**, instead of being tapered as illustrated in earlier rotor assemblies illustrated herein, and in a manner so that they fit within 0.010" of the stator rings **502** shown in FIG. **31** when rotating in a cooperative manner. The purpose of the configuration illustrated in FIG. **27** is to introduce a high number of shears per revolution of the rotor for the energetic mixing and dispersion of liquid solutions as well as powder/

liquid solutions. It would typically be used by the chemical industries and those industries which must intimately and thoroughly mix and disperse materials in their manufacturing processes, such as inks, pigments, dies, powders, etc.

Turning now to FIG. **10** we see a partial cross-section view of a non-intermeshing stator-rotor assembly **100** which has been developed for the effective reduction of friable material to a micron particle size. Friable material will typically shatter upon impact with another object. This principle applies whether the material is impacted from an outside source or from attrition with another similar material. This non-intermeshing stator-rotor configuration of the type illustrated in FIGS. **13** and **14**, comprising a segmented stator assembly **102** and segmented rotor segmented **104**, effectively uses the impact principle of operation by forming a conically shaped coarse grinding cavity **106** seen in FIG. **11** between the two assemblies at the center or sloped portion of the stator-rotor assembly **100** and a fine grinding portion **108** seen in FIG. **12** at the assembly's planar outer or perimeter portion. The basic pie shape of the segments, as further detailed in FIGS. **13** and **14**, are produced in base metal such as 4340 alloy steel and the segments are then overlaid with a tungsten carbide or diamond chip matrix to provide a wear resistant surface. The fine grinding section of the stator-rotor assembly **100** is formed by the planar or outer perimeter portion of the rotor assembly **104** running in close proximity and essentially parallel to the outer perimeter portion of the stator assembly **102**. The corresponding gap **110**, seen enlarged in FIG. **11** between the stator and rotor sections **102,104**, ultimately determines the maximum particle size that can be released from the grinding chamber **11** through port **22**. Therefore, the roughness height should be established at between 32 and 500 micro-inches. It should be noted that the fine grind ring **54** illustrated in FIG. **7** is not necessary with the fine grinding stator-rotor assembly **100**, therefore a blank ring **55** is used in its place. FIG. **11** also shows a typical method of constructing the fine grinding portion **110** of the stator-rotor assembly **100** by overlaying opposing face portions of each segment of the rotor and stator **102, 104** with hard surfacing material in a matrix, such as smooth tungsten carbide or diamond dust **112**, and then grinding it flat with a diamond grinder so that both stator and rotor surfaces run true relative with each other. An alternate means of constructing the fine grinding section **108** is shown in FIG. **12** where tungsten carbide or diamond chips **114** are overlaid on the surface to form a multitude of randomly shaped teeth. In operation, the corresponding rough surface produced on the stator and rotor segments in both the conical cavity **106** and the find grind section **108** effectively reduces material passing through the rotating rotor-stator assembly **100** to micron size particles.

FIG. **14**, illustrating the segmented stator **102**, and FIG. **15**, illustrating the segmented rotor **104**, show this non-intermeshing stator and rotor configuration from a perspective view. The sloping portion **106** of each segmented portion of the rotor and stator **102,104** is shown to have its surface overlaid with tungsten carbide or synthetic diamond chips **112,114**. The size of the chips decreases as the distance from the center of the grinding chamber increases. For example, the larger chips **116** positioned around the center portion of the chamber may range in size from 2 to 4 grit. The next section of medium chips **118** may range between 6–8 grit while the outermost fine chips **120** in the planar or outer perimeter portion may vary in size between 20–1000 grit. The hardness of these materials ranges from 89–94 Rockwell A for Tungsten carbide and 2400–2600 Knoop for diamond chips. The major drawback with this approach lies with the coarse tungsten carbide chips which may be liberated from their matrix during operation of the grinder. When chips are liberated, they must be reduced and pass through the fine grinding section in order to exit from the grinder. This causes undue wear on the stator and rotor segments **102, 104**.

An alternate means of configuring the non-intermeshing stator-rotor assembly is shown in FIG. **15**. This view shows the cross section of the stator and rotor assembly **200** wherein each is overlaid with finely ground tungsten carbide or synthetic diamond chip matrix in the range of 80 to 100 mesh. In this case, grinding and crushing means is provided by the shape of the stator and rotor segments **202,204** themselves. Each stator and rotor assembly **202,204** has one or more segments which form a "rise" or "hill" in the conical grinding chamber **206** as seen in FIGS. **16** and **17**. When at least two segments without a sloping portion **208** of the rotor aligns with the corresponding non sloping portion **208** segments of the stator, a close clearance results so that any material trapped between the two "non-sloping segments" gets crushed. It is not necessary that the "non-sloping faces" be parallel to one another and a preferred configuration would feature a tapered interface (or gap) between at least a portion of the rotor and stator faces so that material would be progressively sized as it traveled further from the center of the grinder.

FIGS. **16** and **17** better illustrate the actual construction of this alternate configuration. In these views, the stator and rotor are both equipped with two high points **208** which come in close proximity to high points on the opposing segment. The conically shaped chamber on the interior of the stator and rotor provides space for larger particles to enter into the chamber before being crushed. The fine grinding perimeter **210** of the segments may be additionally slotted **212** as shown in FIG. **18** to increase the throughput of the machine. The major advantage of this alternate approach is to minimize the particle size of tungsten carbide or synthetic diamond that must pass through the chamber in the event that it is liberated from its substrate matrix during operation. Also, experience has shown that the substrate matrix wear rate is much reduced when using 80 to 100 mesh or grit chips compared to coarsermesh or grit chips because less of the matrix is exposed to erosion and wear.

Another interchangeable non-intermeshing rotor-stator assembly configuration suitable for refining fibrous materials is shown in FIGS. **19** and **20**. This configuration features segmented stator **300** and rotor **302** having surfaces running parallel and in close proximity to one another. The face of both the stator and rotor segments **304, 306** has a series of parallel grooves **308** cut into the surface at obtuse angles to provide a multitude of channels in which a fibrous slurry may travel and be dispersed without causing undue breakage of the fiber length. Segments are typically made from hardened alloy steel. Such non-intermeshing segmented stator assemblies are typically mounted as seen in FIG. **9**, where the stator plate **44** is secured to the cover plate **18** with bolts **19**. The segments **304** of the stator assembly **300** are located on the face of the stator plate by dowels **41** and secured to the stator plate **44** with bolts **45**. It should be noted that the fine grind ring **54** may be used with the stator assembly **300** and that the fine grind ring **54** may be segmented with such segments **51** located on the face of the stator plate by dowels **49** and secured to the stator plate **44** with bolts **53**. A similar arrangement is seen in FIG. **23** for the rotor assembly **302**, rotor plate **46**.

A more detailed view of the unique mechanical seal cartridge assembly **70** is provided in FIG. **29**. We see the cartridge **70** is comprised of a double pair of mechanical seals **83** imposed on a shaft sleeve **85**, both of which are retained within the cartridge housing **99** with a flange cover **87**. It is also obvious that a single mechanical seal **83** could be arranged for use in some cases. The sleeve **85** is secured to the rotor shaft by the shaft collar **103** containing several set screws **105** which pass though the sleeve **85** and impinge the shaft **50**. As seen in FIG. **30**, the cartridge assembly **70** is attached to the end of the quill by bolts **107** passing though the flange cover **87** and cartridge housing **99**. O-rings **77** positioned around the perimeter of the mechanical seal housing **99** and the quill **48** seal the quill assembly **47** in the longitudinal bore of the casing **12** so that process fluid circulating inside the grinding chamber **9** is prevented from leaking into the longitudinal bore of the casing **12**. The seal housing **99** includes a beveled or lip portion **101** which projects into the longitudinal bore of the quill **48** and is sized to fit against the outer race of the shaft thrust bearing **68** without interference with the lock nut **57** and lock washer **59**. With this configuration, the mechanical seal cartridge assembly **70** is used to clamp the thrust bearing **68** outer race against the shoulder **109** of the bearing cavity or pocket formed in the longitudinal bore of the quill **48**, thereby forcing the shaft **50** and entire quill assembly **47** to move as a single unit. As seen in FIG. **30**, using a mechanical seal assembly **70** to secure a portion of the thrust bearing **68** into its cavity within the slidable quill **48** results in minimizing the distance between the thrust bearing **68** and the rotor end of the shaft **50**, thereby reducing shaft flexure and run-out. This configuration also improves the longevity of the mechanical seal and thus insures stator-rotor assembly alignment, which is essential with variable displacement stator/rotor assemblies.

As stated above, the mechanical seal **70** may be provided with a single or a double pair of mechanical seals. It is also possible to provide packing inside the seal cartridge **70** as a means of sealing the shaft in lieu of mechanical seals. However, the preferred embodiment of the seal cartridge **70** includes the use of double mechanical seals **83** subjected to a barrier fluid circulated into and out of the seal cavity through a pair of ports **88** indicated in FIG. **30**. The barrier fluid is typically delivered at a pressure of 15 to 20 PSI above the process pressure inside the grinding chamber **9** as a means of insuring that the faces of the mechanical seals **83** are always lubricated and cooled by the barrier fluid. Process fluid would enter between the mechanical seal faces if the barrier fluid were not pressurized. Circulation of the barrier fluid through the seal **83** further provides an opportunity to continuously remove heat from the seal while simultaneously lubricating the seal faces. A single pair mechanical seal would not require the use of a barrier fluid, however, packing would perform best when continuously flushed and cooled with an external fluid source. However, the barrier fluid may tend to leak into the process fluid.

Having fully described the many options for configuring the in-line grinder, it has been found that this type of apparatus is directly applicable to the reduction of drill cuttings for injection into a wellbore formation. As a matter of background, a wellbore is formed in a generally conventional manner by providing a wellhead for supporting a casing string which extends within the wellbore. A drive pipe extends into the formation in support of the wellhead. Cement occupies the annular space between the drive pipe and the casing as well as an annular area between the formation and the casing. A secondary casing or protection pipe extends from the wellhead into the formation and is cemented at a zone which has been packed with cement and which leaves an annular area or space between the cement and the casing which is delimited by the formation and the protection pipe. A drill stem typically extends through the wellhead, the casing and the protection pipe to an open hole bottom portion of the well-bore. In accordance with conventional drilling practice, drilling fluid is circulated from a source down through the drill stem and up through the annular area formed between the drill stem and the pipe to a return receptacle or bell nipple. The drilling fluid returning through the annulus carries with it the earth particles or drill cuttings which, upon return to the surface, are conducted by way of a conduit to a separating device commonly known as a shale shaker. Drill cuttings which are too large to be included in the drilling fluid for recirculation into the wellbore are separated by the shale shaker and conducted by suitable conduit means to a unique system for treating and disposing of the drill cuttings in accordance with the present invention. Drilling fluid and finer drill cuttings particles not separated by the shale shaker are collected in a mud tank and processed in accordance with conventional practices before reinjection of the drilling fluid down through the drill stem. Smaller drill cuttings not separated by the shale shaker may be separated in conventional desanders and added to a slurry to be described herein.

In accordance with the present invention, a unique system is provided for processing the separated drill cuttings into a homogenous mix prior to injection into the earth formation. FIG. **21** illustrates a configuration of the system **400** in schematic form. The system **400** includes a receiving or slurry tank **402** which is fitted with a suitable agitation device **404**. The slurry tank **402** is in fluid communication with a shale shaker **406**, usually located on the drilling platform producing the drill cuttings **408**, by way of a conveyor **410** for receiving drill cuttings **408** from the shale shaker **406**. The slurry tank **402** is also in fluid communication with a conduit **412** which is connected to a source of slurry carrier liquid, which may be sea water **414**, fresh water **416**, or waste water **418** from the platform's sewage treatment system. A separate viscosity enhancing polymer line **420** is also routed to the slurry tank **402**. The system **400** also includes one or more transfer pumps **422** which are in fluid communication with the slurry tank **402** by way of suction lines **424**, **426**. Positioned between the slurry tank **402** and transfer pump(s) **422** is a means of removing tramp metal and other unprocessable items, such as a magnetic trap **428**. The transfer pump(s) **422** delivers the drill cuttings slurry **408** to one or more properly configured in-line grinder(s) **430** discussed herein for sizing the solids prior injection into the formation using a high pressure pump (usually a positive displacement pump). Valves **434** are provided for directing the sized cuttings either directly to the injection pump or back into the slurry tank **402**. This option allows the system to be operated in a continuous fashion, in a batch mode, or a hybrid mode. In the continuous mode, drill cuttings **408** are continuously received, sized, conditioned, and delivered to the injection pump. In the batch mode, drill cuttings are received on an intermittent basis and recirculated through the in-line grinder until a tank size quantity of material is properly sized and conditioned. Afterwards, it is directed to the injection pump. Finally, the hybrid mode involves the continuous receipt of drill cuttings **408** and the recirculation of those drill cuttings through the in-line grinder **430** and back into the slurry tank **402**. A side stream is continuously extracted from the discharge of the in-line grinder **430** and it is routed to the injection pump.

The sized solids leaving the in-line grinder **430** are suitable for routing through a mass flow meter **432** for the purpose of generating a signal proportional to the density of the slurry **436**. This signal is input into a process controller **438** which modulates the flow of water into the slurry tank through a control valve **440** installed on the water input line **412** to the tank **402**. This control loop provides a continuous means of delivering a constant density slurry to the formation. Further, the slurry tank **402** is equipped with a viscosity transmitter **442** which produces a signal proportional to the viscosity of the slurry **436**. This signal is input into a process controller **444** which modulates the flow of viscosity enhancing polymer **420** into the slurry tank **402**. The net result of the viscosity and density control systems is to deliver a sized slurry to the formation which has consistent and ideal properties for effective migration throughout the formation without plugging it. An optional means of introducing dilution water and viscosity enhancing polymers to the drill cuttings is by injecting them into the suction line of the in-line grinder **430** via control valve **446**. The in-line grinder **430** has the ability to instantaneously disperse and grind the slurry so that quick adjustments can be made automatically to vary the slurry properties.

FIG. **22** illustrates the preferred embodiment of the drill cuttings processing system. It features a slurry system **500** comprising a slurry tank **402**, transfer pump(s) **422,506**, in-line grinder(s) **430**, piping, and instrumentation as outlined above. The system additionally comprises an injection system which receives the sized and conditioned drill cuttings from the slurry system. The injection system comprises an agitated injection tank **502**, as well as one or more transfer pumps **506**, piping, and instrumentation. The transfer pump **506** takes processed cuttings slurry **504** from the injection tank **502** and directs it to the injection pump; recirculates it back to the injection tank **502**; directs it to the suction side of the in-line grinder(s) **430**; or directs it to the slurry tank **402**. Also piping interconnections are provided between the slurry tank transfer pump(s) **422**, and the injection tank transfer pump(s) **506** so that each may operate from either tank, thereby increasing the versatility of the system. The injection tank **502** is also equipped with a fine viscosity transmitter **508** which delivers its signal to a fine viscosity controller **510**. The viscosity controller **510** modulates the flow of viscosity enhancing polymers **420** into the injection tank **502**. In like manner, the injection tank transfer pump **506** routes its flow through a second mass flow meter **432** for the purpose of generating a signal proportional to the density of the cuttings slurry **504**. A fine density controller **512** receives the signal from the mass flow meter **432** and modulates the flow of dilution water **414–118** into the injection tank **502**.

In normal operation, drill cuttings **408** are continuously sized and conditioned by the slurry system **500** and held in the injection tank **502** for injection. The density and viscosity adjustments made to the drill cuttings in slurry **436** is generally coarse in nature due to the variations in drill cuttings delivered to the system. The injection tank **502**, being equipped identically as the slurry tank **402**, has the ability to make fine adjustments to the properties of the drill cutting slurry **504** before injection into the formation by the injection pump. Therefore, the consistency and quality of the drill cuttings may be improved through the use of this automatic dual adjustment system **500**. Flow control of cuttings to the formation may be regulated through variable speed control of the injection pump or through the use of a control valve **514** to bypass excess flow back to the injection tank.

The method and system of the present invention described herein above provides a simplified way of disposing of earth drill cuttings heretofore unappreciated within the art. Although a preferred embodiment of a method and a system structure, provided in a accordance with the present invention have been described herein above, those skilled within the art will recognize that various substations and modifications may be made to the specific embodiments described without departing from the scope and spirit of the invention as recited in the appended claims.

What is claimed is:

1. A mechanical seal and thrust bearing assembly for use in variable displacement slurry type rotary grinders to prevent shaft deflection and run-out comprising:

a) a housing wear sleeve;

b) a mechanical seal cartridge including a bearing housing and a seal housing both being slidable relative to said wear sleeve; and

c) an inner race having shaft compression lock and sealing means said race extending longitudinally through said seal housing.

2. The mechanical seal and thrust bearing assembly according to claim **1** further comprising:

a) a removable flange having a stationary seal therein, attachable to one end of said bearing housing for capturing said seal housing mesial said flange and said bearing housing, said seal housing having a protruding portion at one end opposite said flange;

b) a thrust bearing located within said bearing housing the outer race of which is in contact with said protruding portion;

c) a means for securing said flange to said bearing housing;

d) at least one mechanical seal assembly located within said seal housing and in rotational contact with said inner race;

e) a means for sealing said seal housing relative to said mechanical seal and said bearing housing; and

f) a means for sealing said seal housing relative said housing wear sleeve.

3. The mechanical seal and thrust bearing assembly according to claim **2** further comprising a means for sealing said housing wear sleeve relative to a removable rotor housing.

4. An improved variable displacement rotary grinder the improvement comprising:

a) a rotary grinder of the type having a variable displacement rotor/stator assembly wherein said rotor is driven by a displaceable shaft attached perpendicular to said rotor extending through a shaft housing; and

b) a mechanical seal cartridge assembly including a thrust bearing removably secured to said shaft and located mesial said rotor/stator assembly and means for displacing said shaft, said cartridge being slidable within said housing.

5. The variable displacement rotary grinder according to claim **4** wherein said mechanical seal further comprises:

a) a seal housing having a removable cover portion and a protruding portion for contacting said outer race of said thrust bearing;

b) a means for securing said cover portion to said seal housing;

c) a seal fixed to said seal housing and rotatable relative to said inner race said inner race extending length of said seal housing and beyond said cover portion;

d) a seal fixed to said cover portion and rotatable relative to said inner race;

e) a means for securing said inner race portion of said shaft;

f) a mechanical seal having at least one seal set including a stationary seal face, a rotating seal face and means for compressing said seal faces, a portion of said at least one seal set being rotatable relative to said inner race, said at least one seal set located within said seal housing; and

g) a means for sealing said seal housing relative said shaft housing and said rotor housing.

6. The variable displacement rotary grinder according to claim **5** further comprising a sleeve rotatable and slidable relative said seal housing located within said shaft housing and having means for sealing relative said rotor housing.

7. A method of minimizing shaft deflection and run out in a slurry type grinder rotor shaft requiring bearing sealing means in the region between a rotor disk and a shaft support bearing comprising the steps of:

a) providing a rotor shaft attachable at one end to a rotor assembly

b) providing and supporting said shaft at least in part by a shaft thrust bearing having inner and outer race and wherein said inner race is secured to said shaft;

c) providing a mechanical seal assembly comprising:

i) a seal housing having a cover member, a protruding portion opposite said cover and external sealing means for sealing said seal housing within a shaft housing;

ii) a means for retaining said cover to said housing;

iii) a hollow sleeve member including a set collar extending centrally through said housing and said cover removably attached to said rotor shaft;

iv) at least one mechanical seal assembly located within said housing and rotatable relative to said sleeve member;

v) a seal member fixed to said cover and rotatable relative to said sleeve; and

vi) a seal member fixed within said seal housing rotatable relative to said sleeve;

d) locating said mechanical seal assembly upon said rotor shaft in close proximity to said a rotor assembly and in contact with said shaft thrust bearings in a manner whereby said protrusion is in compressive contact with said outer race of said thrust bearing.

8. The method according to claim **7** further comprising the step of providing a tubular sleeve having sealing means and positioning said sleeve in a manner whereby said sleeve is in sliding engagement with said external sealing means of said seal housing.

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