US006102385A

United States Patent [19]

Wakamatsu et al.

[11] **Patent Number: 6,102,385**

[45] **Date of Patent: Aug. 15, 2000**

[54] **FINISHER**

[75] Inventors: **Shinji Wakamatsu**; **Norihiko Suzuki**, both of Toyokawa, Japan

[73] Assignee: **Minolta Co., Ltd.**, Osaka, Japan

[21] Appl. No.: **08/946,489**

[22] Filed: **Oct. 7, 1997**

[30] **Foreign Application Priority Data**

Mar. 12, 1997 [JP] Japan 9-058120

[51] **Int. Cl.**[7] **B65H 39/02**

[52] **U.S. Cl.** **270/58.12**; 270/58.08; 270/58.07; 270/58.09; 271/221; 271/222; 271/233; 271/236

[58] **Field of Search** 270/58.12, 58.16, 270/58.08, 58.07, 58.09; 271/221, 222, 233, 236, 239, 245, 226

[56] **References Cited**

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5,905,935	5/1999	Wakamatsu et al.	399/407

Primary Examiner—Joseph E. Valenza
Assistant Examiner—Michael E. Butler
Attorney, Agent, or Firm—Sidley & Austin

[57] **ABSTRACT**

A finisher includes an additional-working tray and a leading edge stopper and trailing edge stopper for aligning folded and unfolded sheets fed into the additional-working tray. The leading edge stopper contacts a leading edge of each sheet when it is fed into the additional-working tray. The trailing edge stopper is movable relative to the additional-working tray in accordance with a sheet length. The trailing edge stopper is adapted to align one or more sheets received in the additional-working tray relative to a discharge direction.

23 Claims, 41 Drawing Sheets

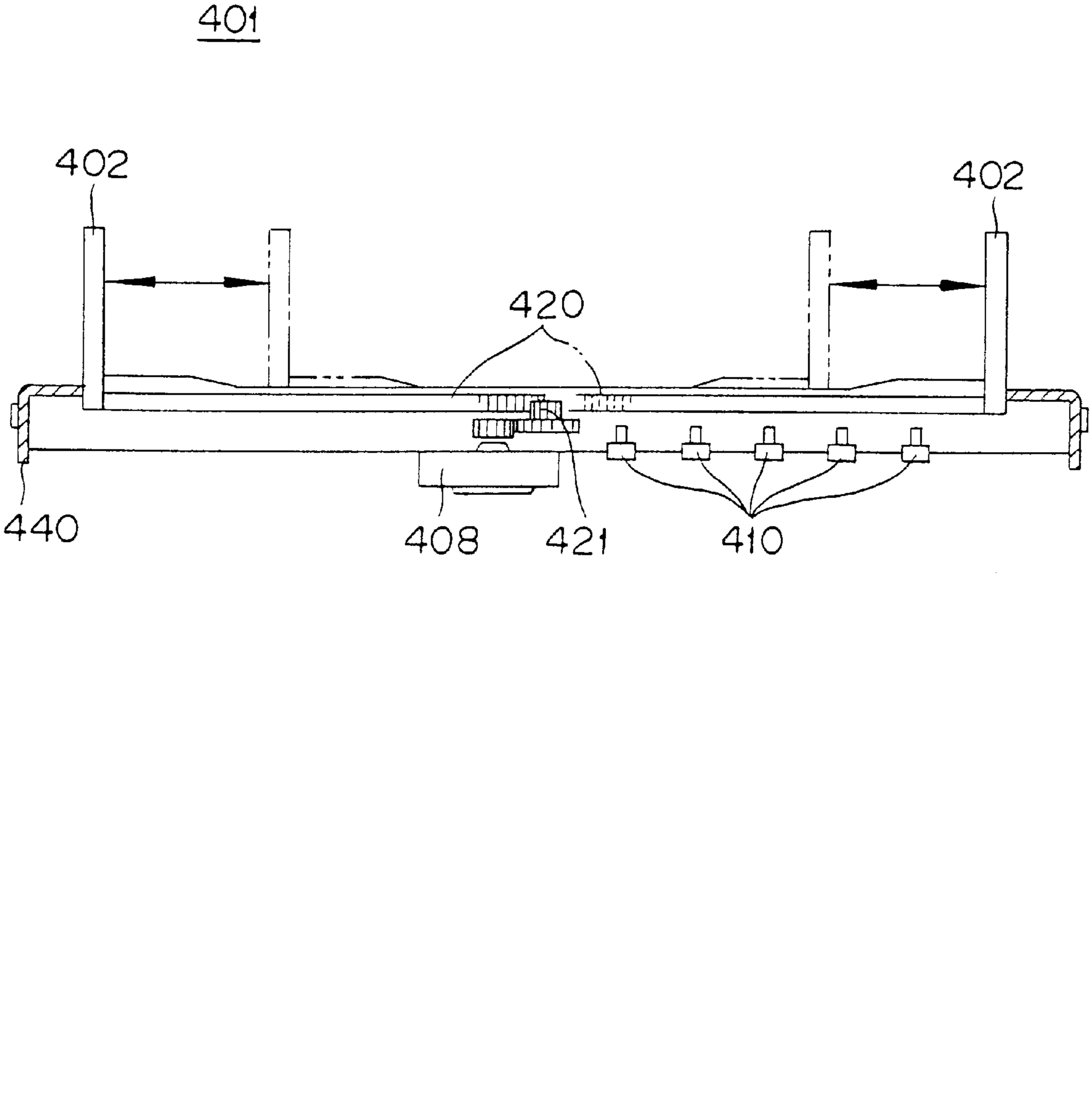


FIG. 1

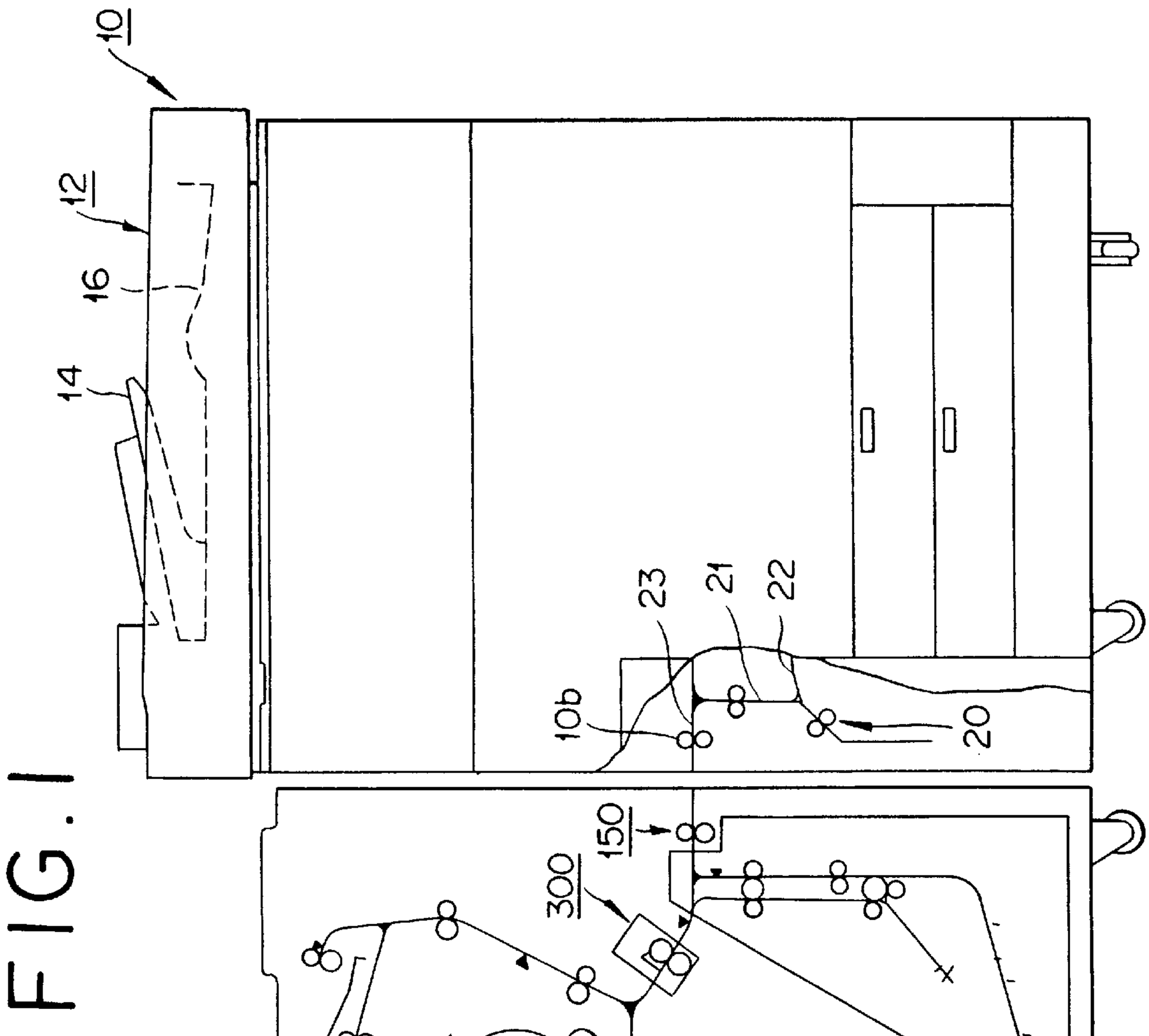
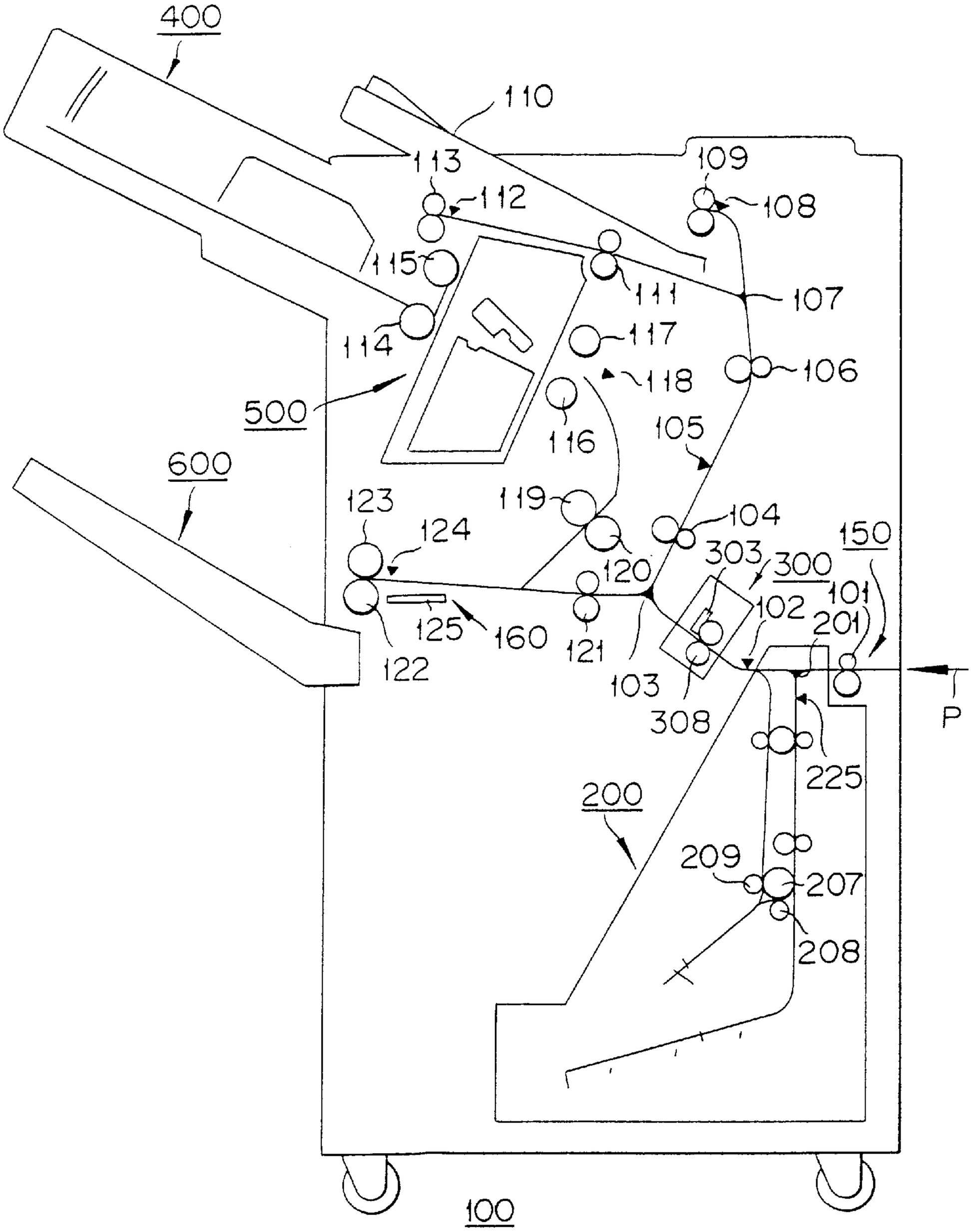


FIG. 2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124
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308
P
225
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209
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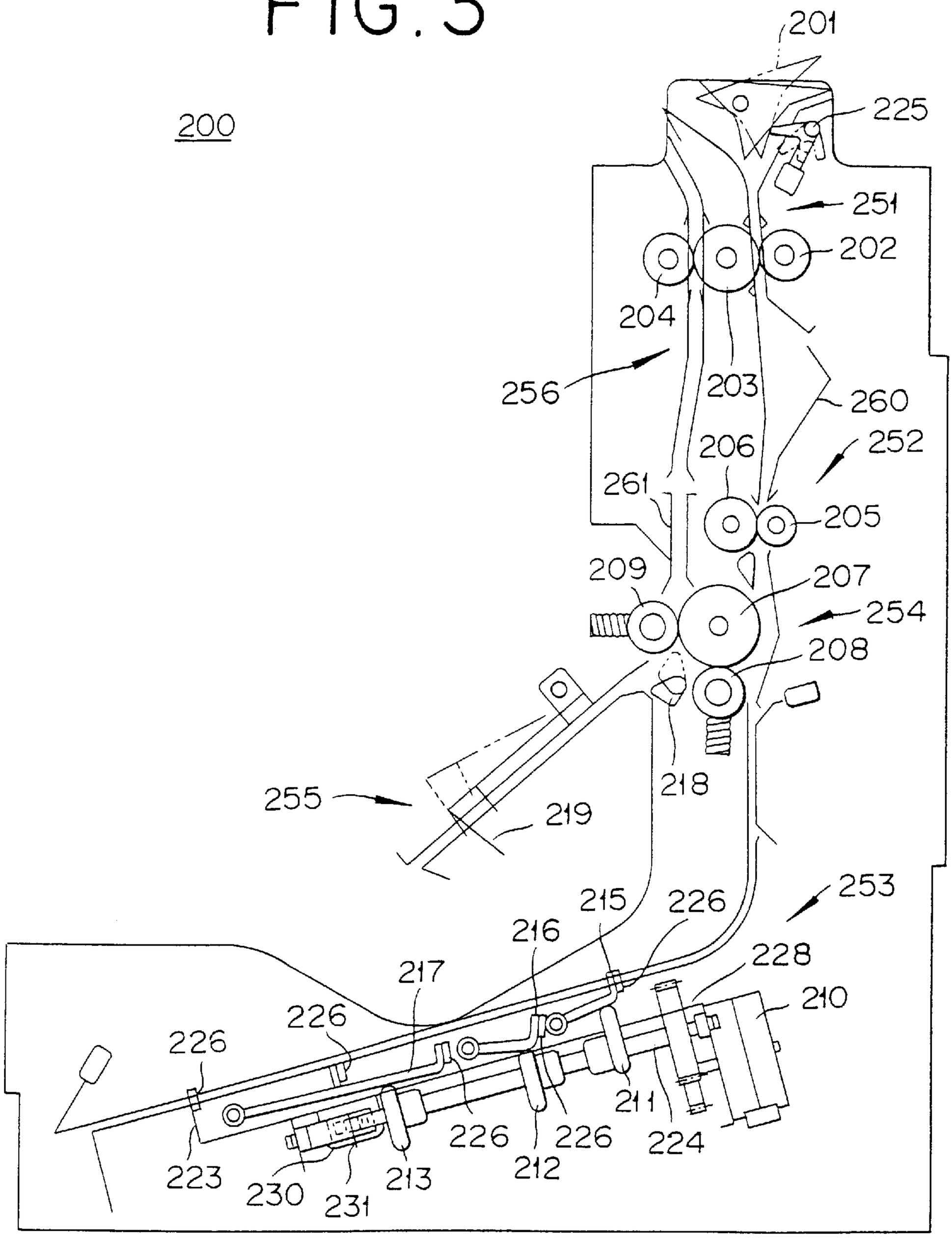
208
100

FIG. 3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213
212

FIG. 4

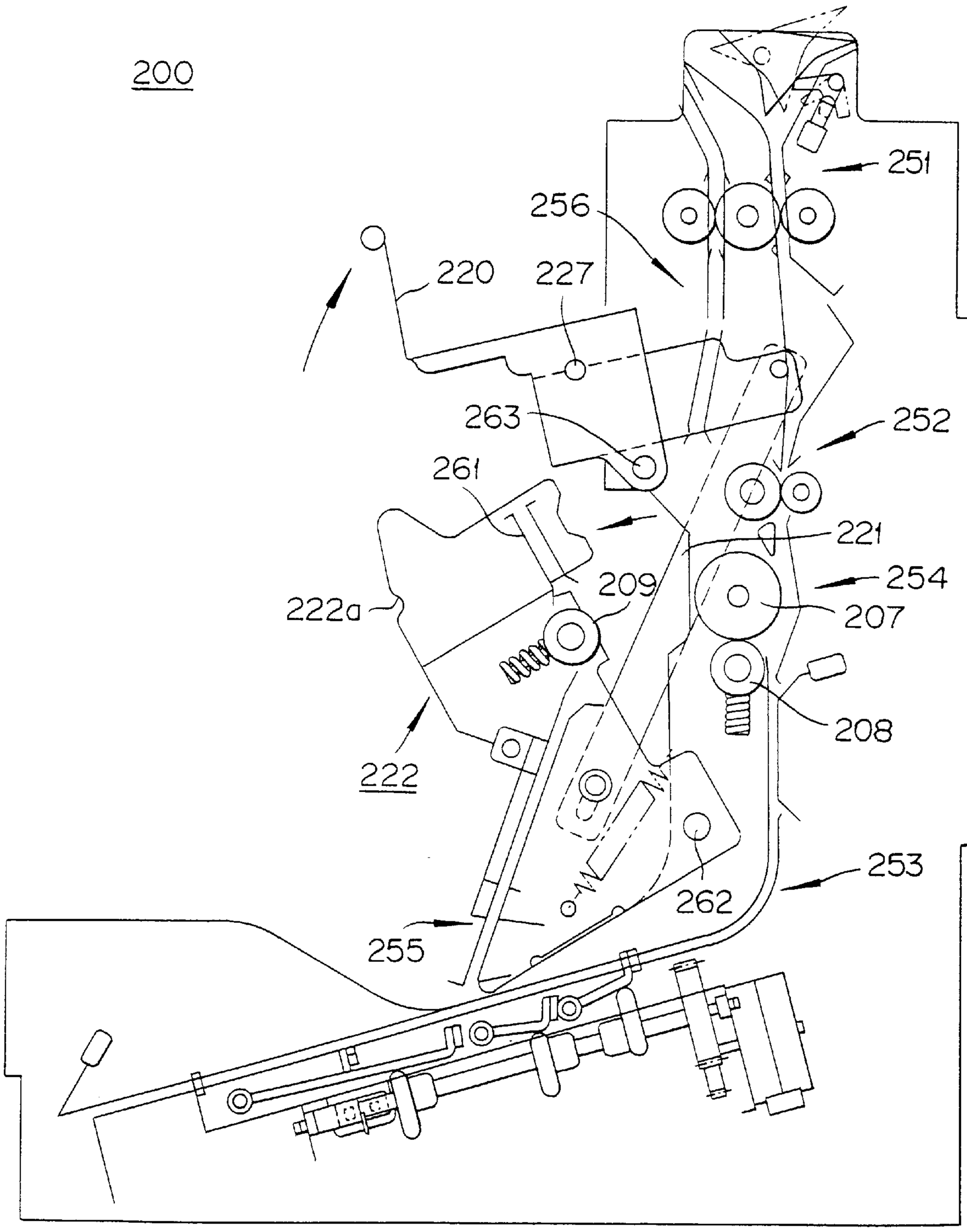

200
251
256
220
227
263
252
261
221
254
209
207
222a
208
222
253
262
255

FIG. 5A

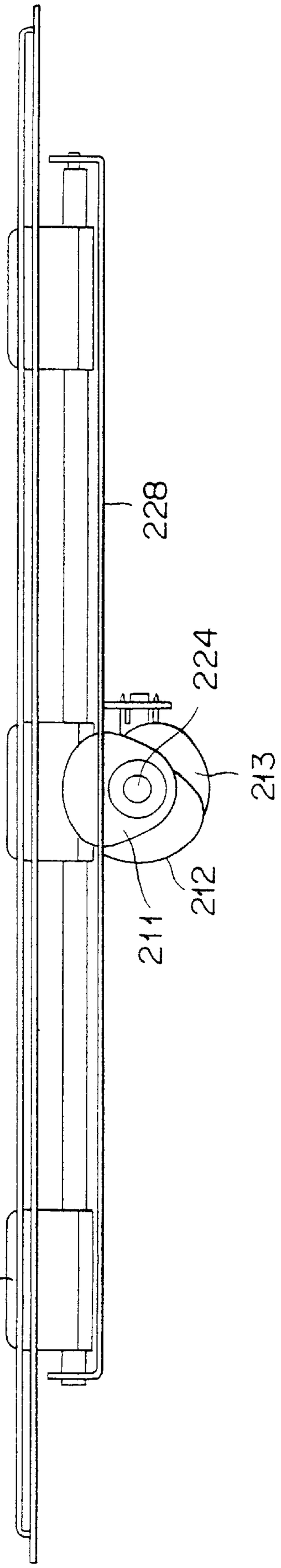


FIG. 5B

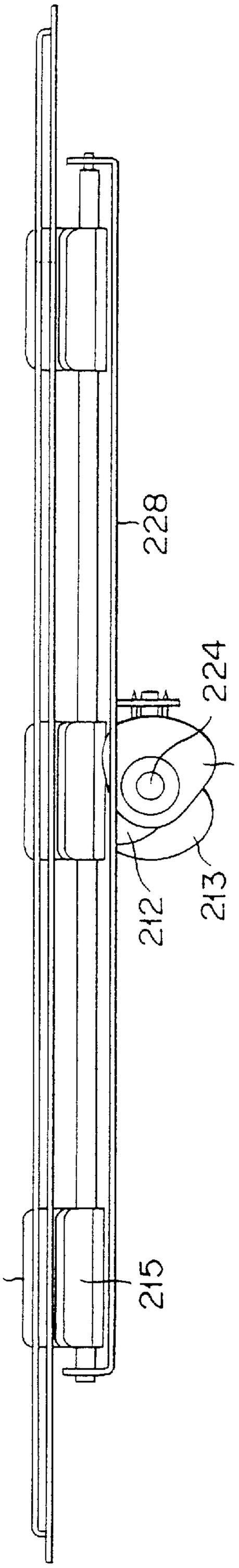


FIG.6

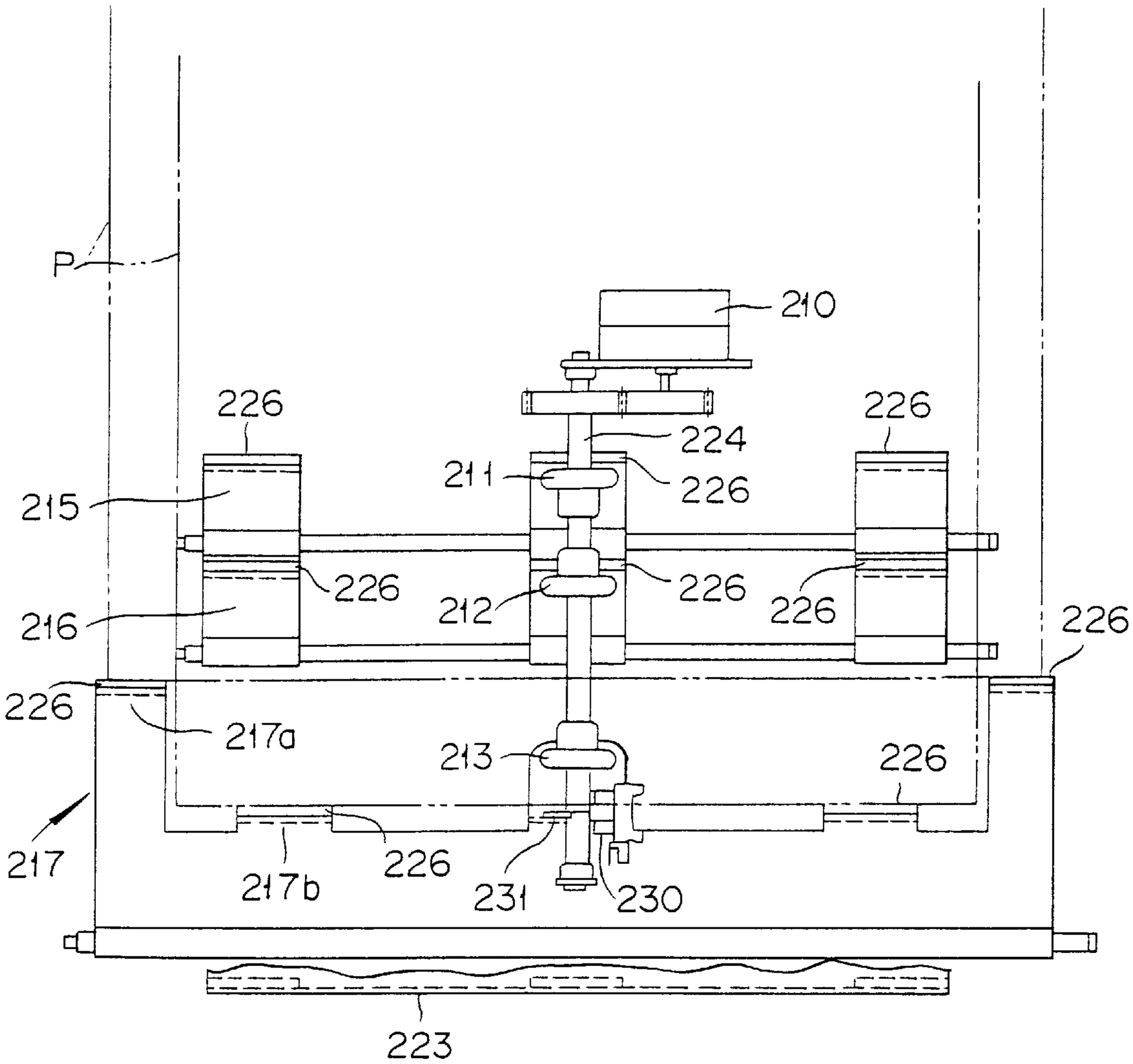

P
226
210
224
211
226
215
226
226
226
212
226
226
216
226
226
217a
213
226
217
217b
226
231
230
223

FIG. 7

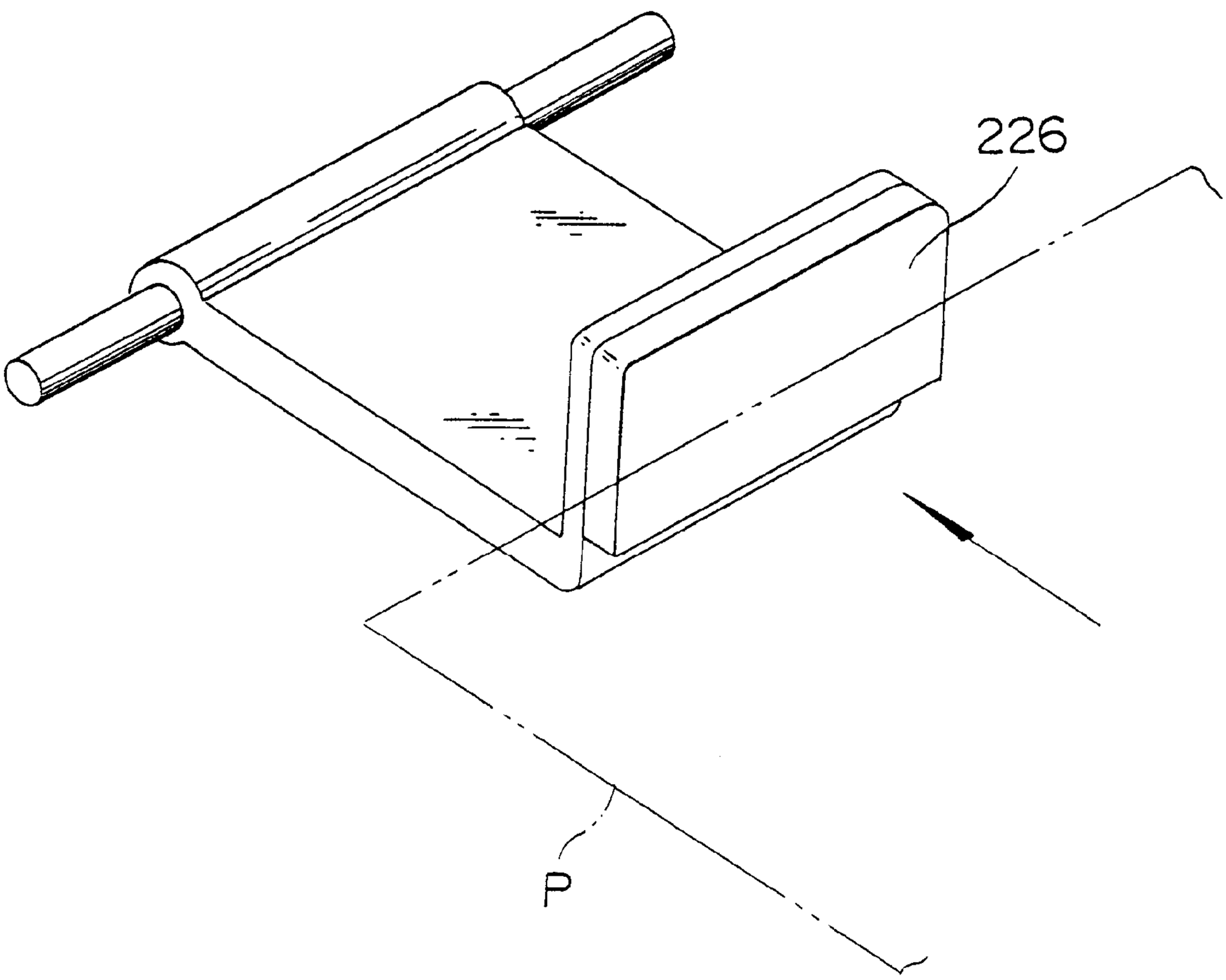
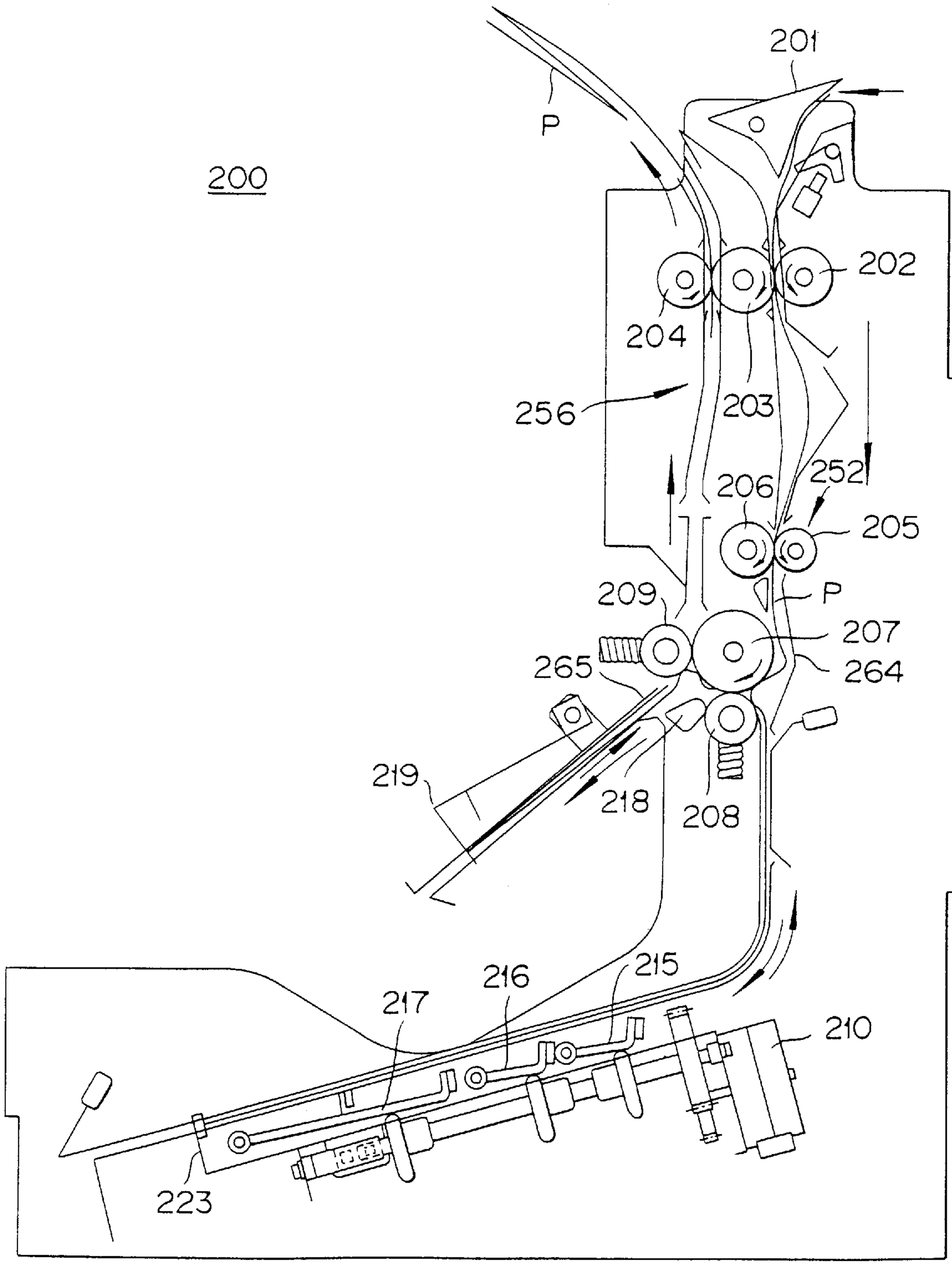


FIG. 8


200
201
P
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204
256
203
206
252
205
209
P
207
264
265
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215
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210
223

FIG. 9

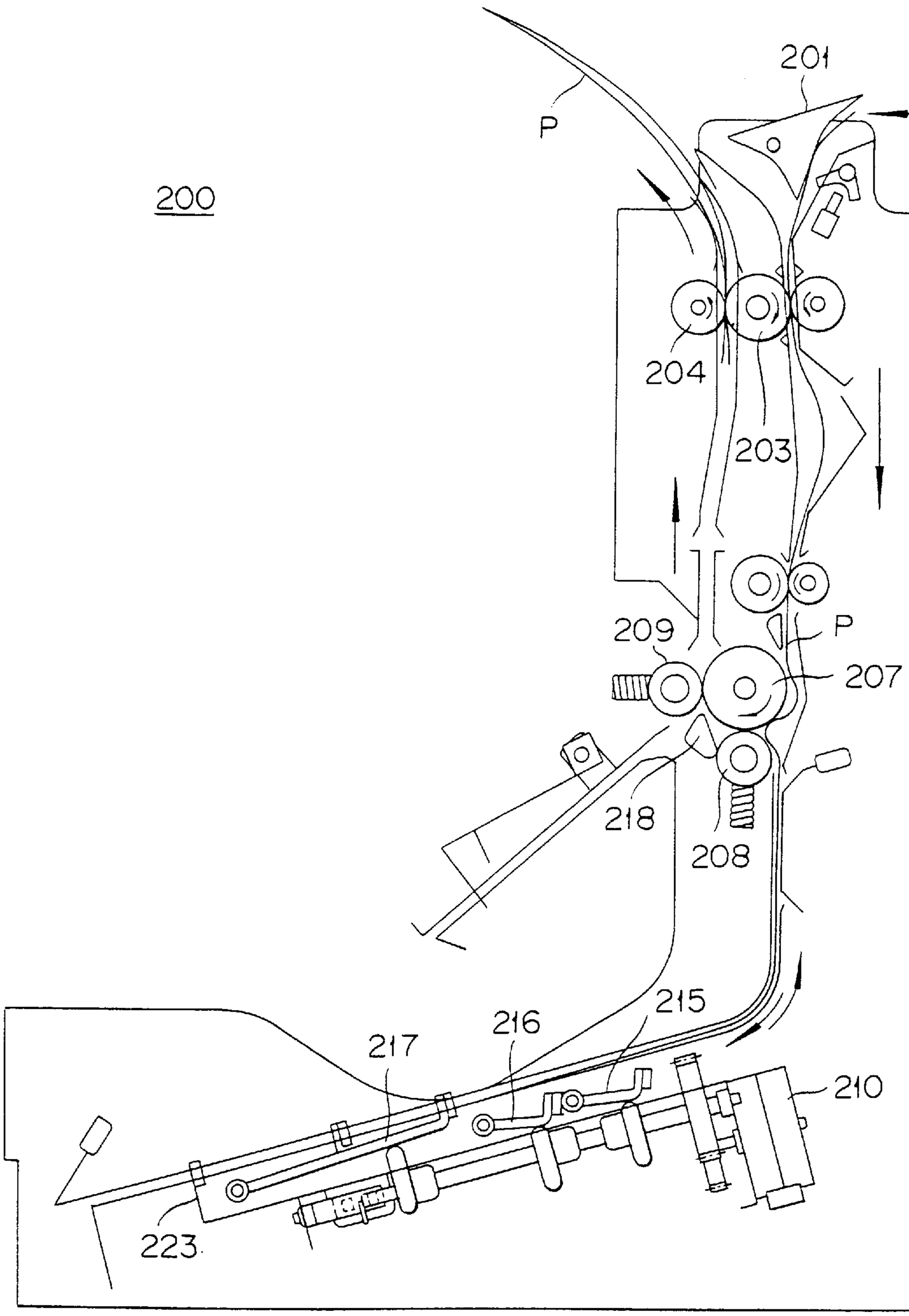


FIG. 10

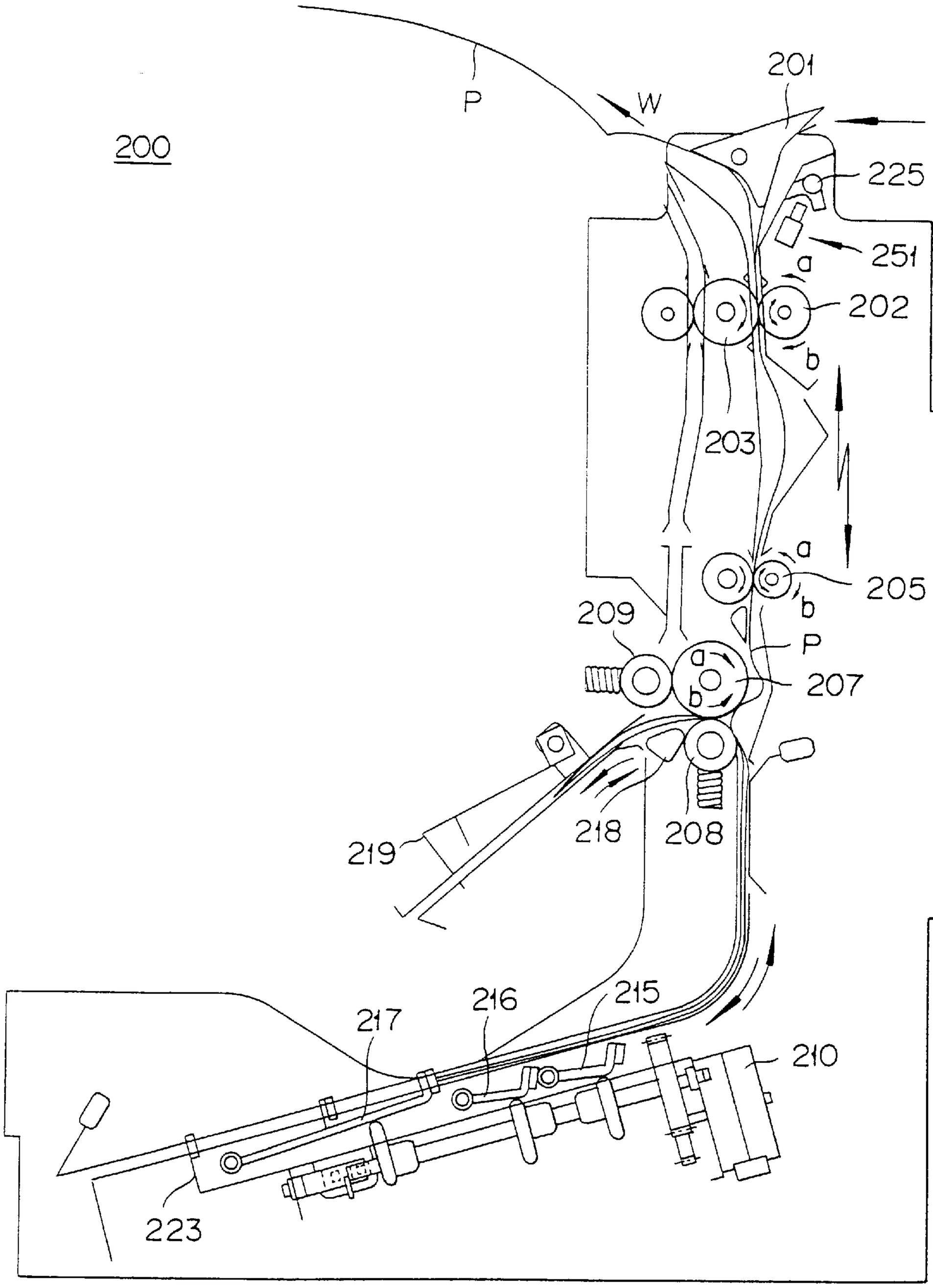

200
P
W
201
225
251
a
202
b
203
a
205
209
b
P
a
207
b
219
218
208
216
215
217
210
223

FIG. 11

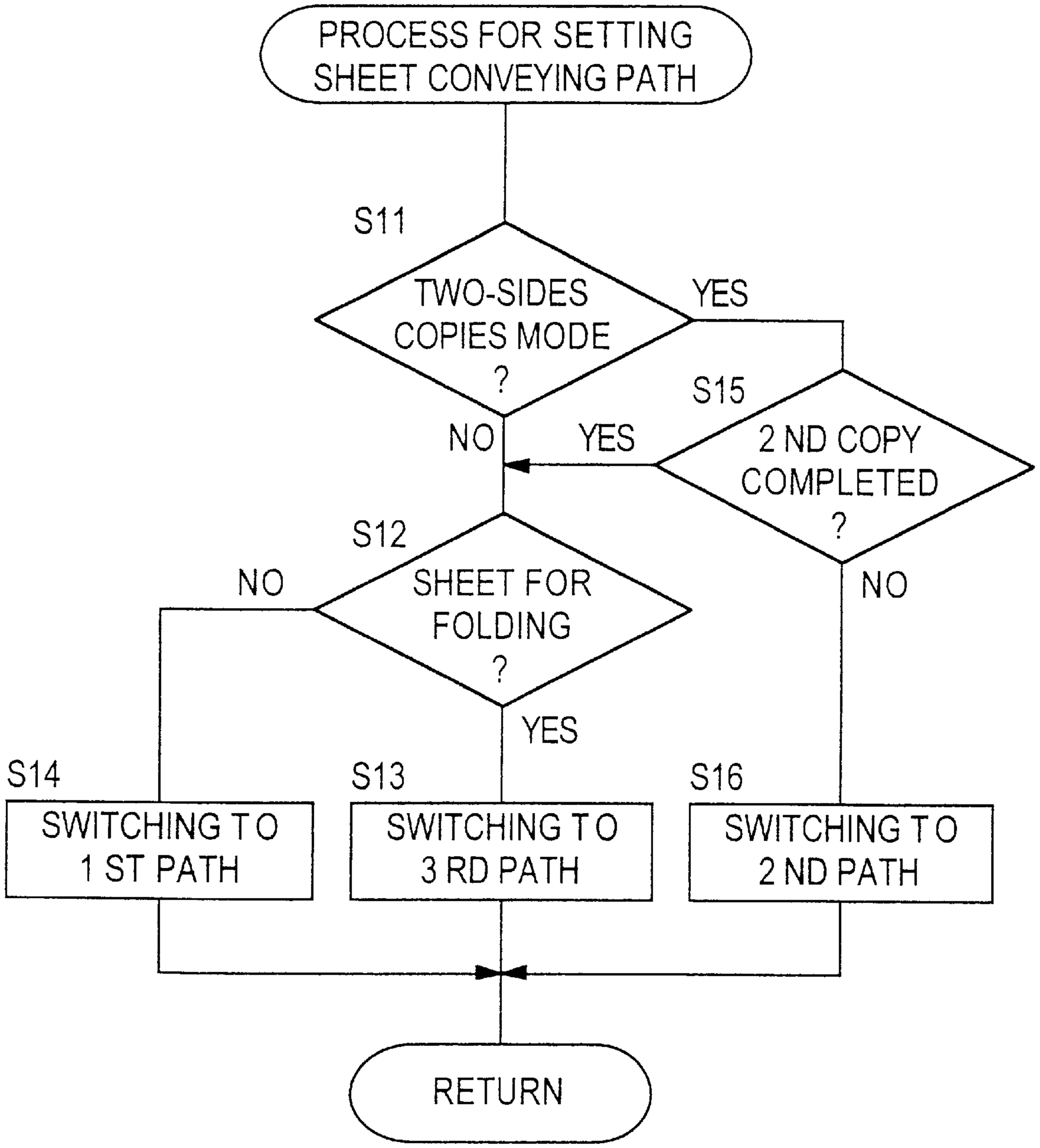
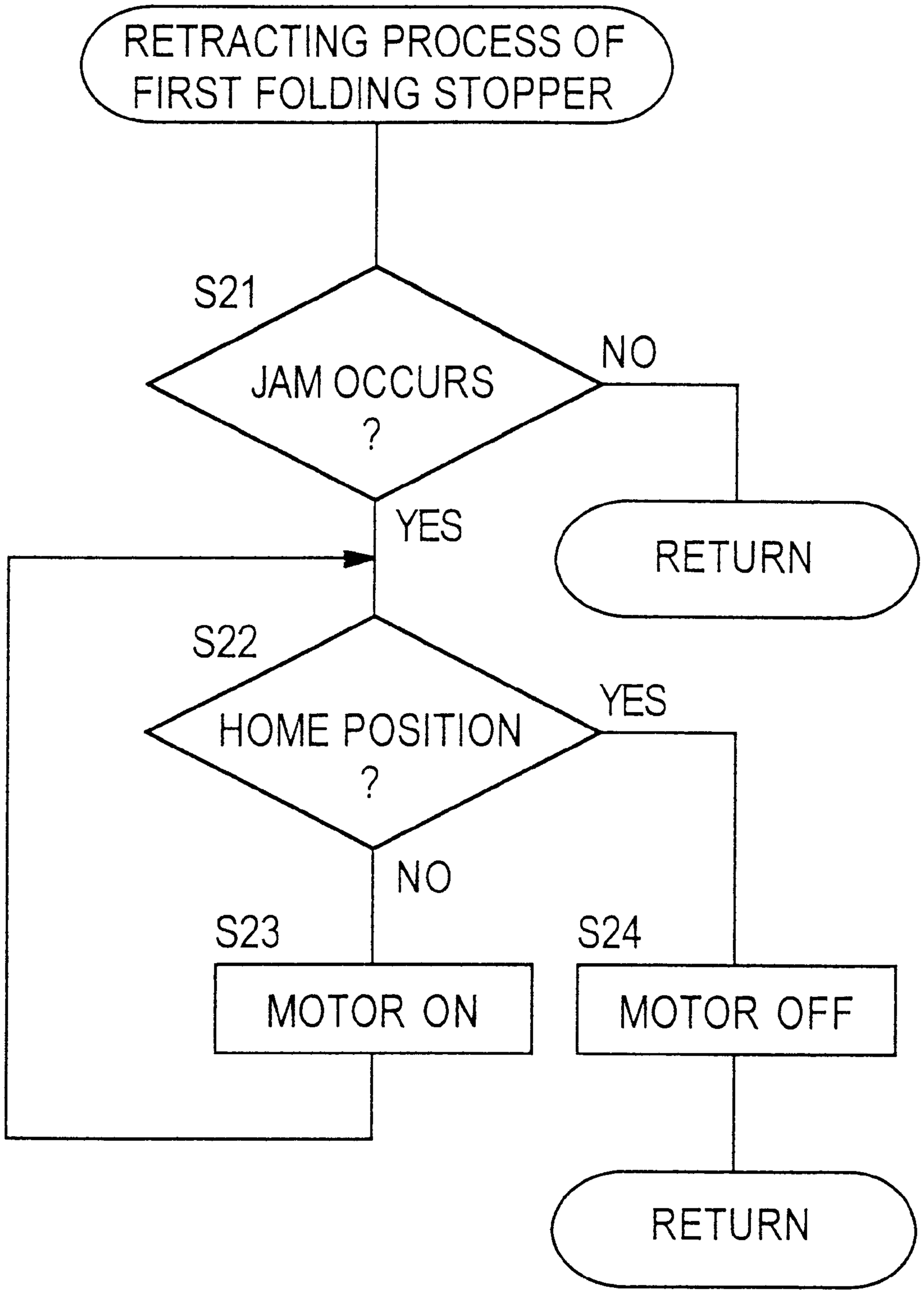

PROCESS FOR SETTING SHEET CONVEYING PATH
S11
TWO-SIDES COPIES MODE ?
YES
NO
S15
2 ND COPY COMPLETED ?
YES
NO
S12
SHEET FOR FOLDING ?
NO
YES
S14
SWITCHING TO 1 ST PATH
S13
SWITCHING TO 3 RD PATH
S16
SWITCHING TO 2 ND PATH
RETURN

FIG. 12


RETRACTING PROCESS OF FIRST FOLDING STOPPER
S21
JAM OCCURS ?
NO
YES
RETURN
S22
HOME POSITION ?
YES
NO
S23
MOTOR ON
S24
MOTOR OFF
RETURN

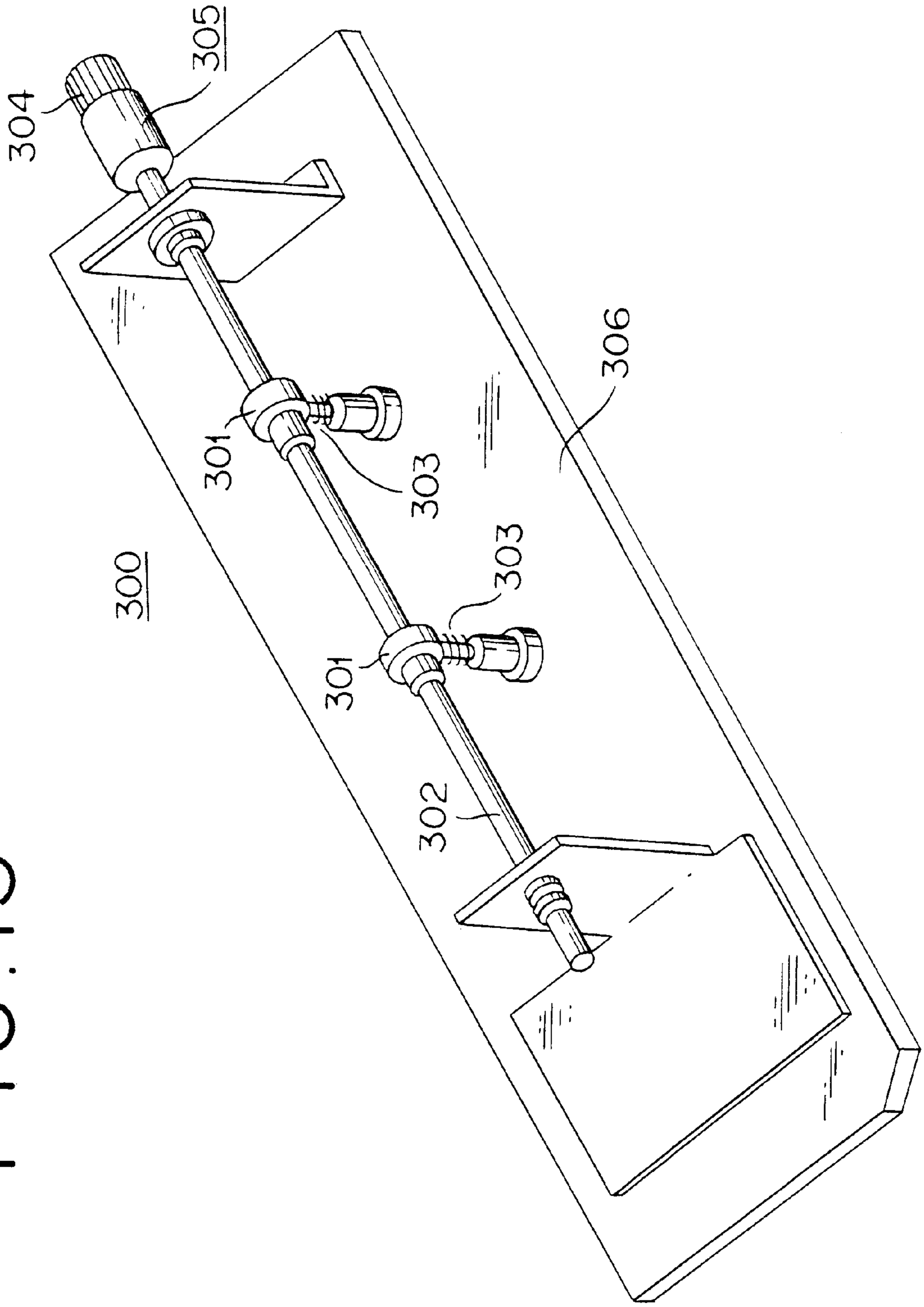


FIG. 13

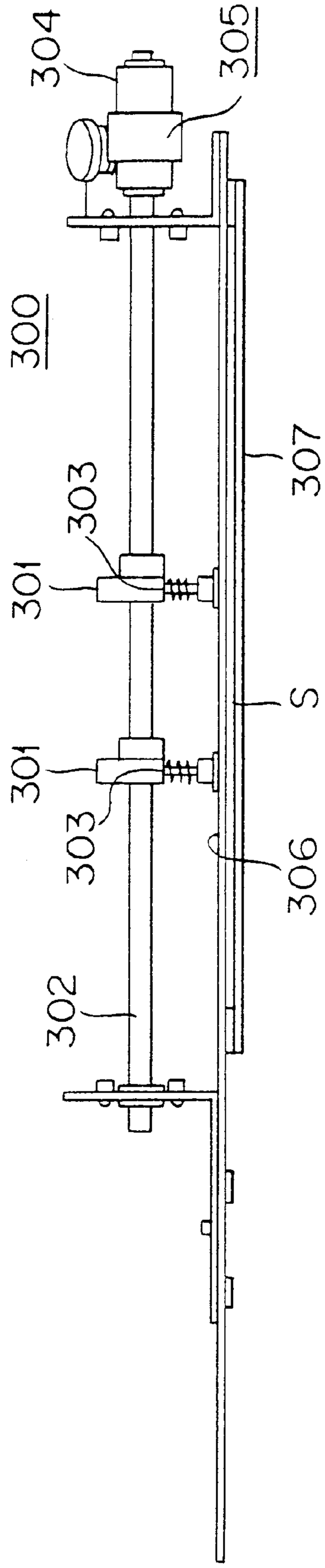


FIG. 14

FIG. 15

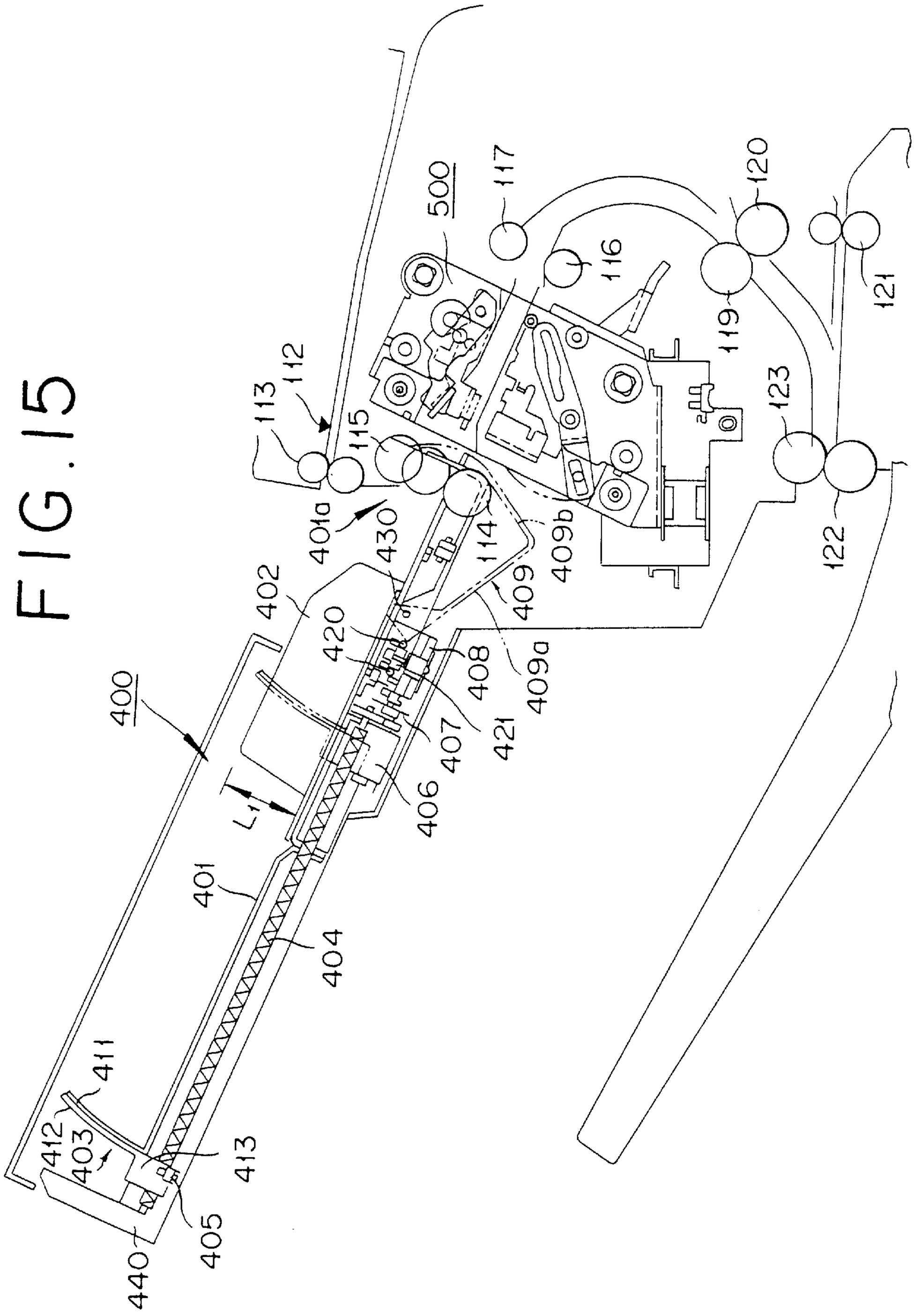

500
117
120
116
121
119
112
113
115
123
401a
430
114
409b
122
409
402
420
408
409a
400
407
421
406
L1
401
404
411
412
403
413
405
440

FIG. 16

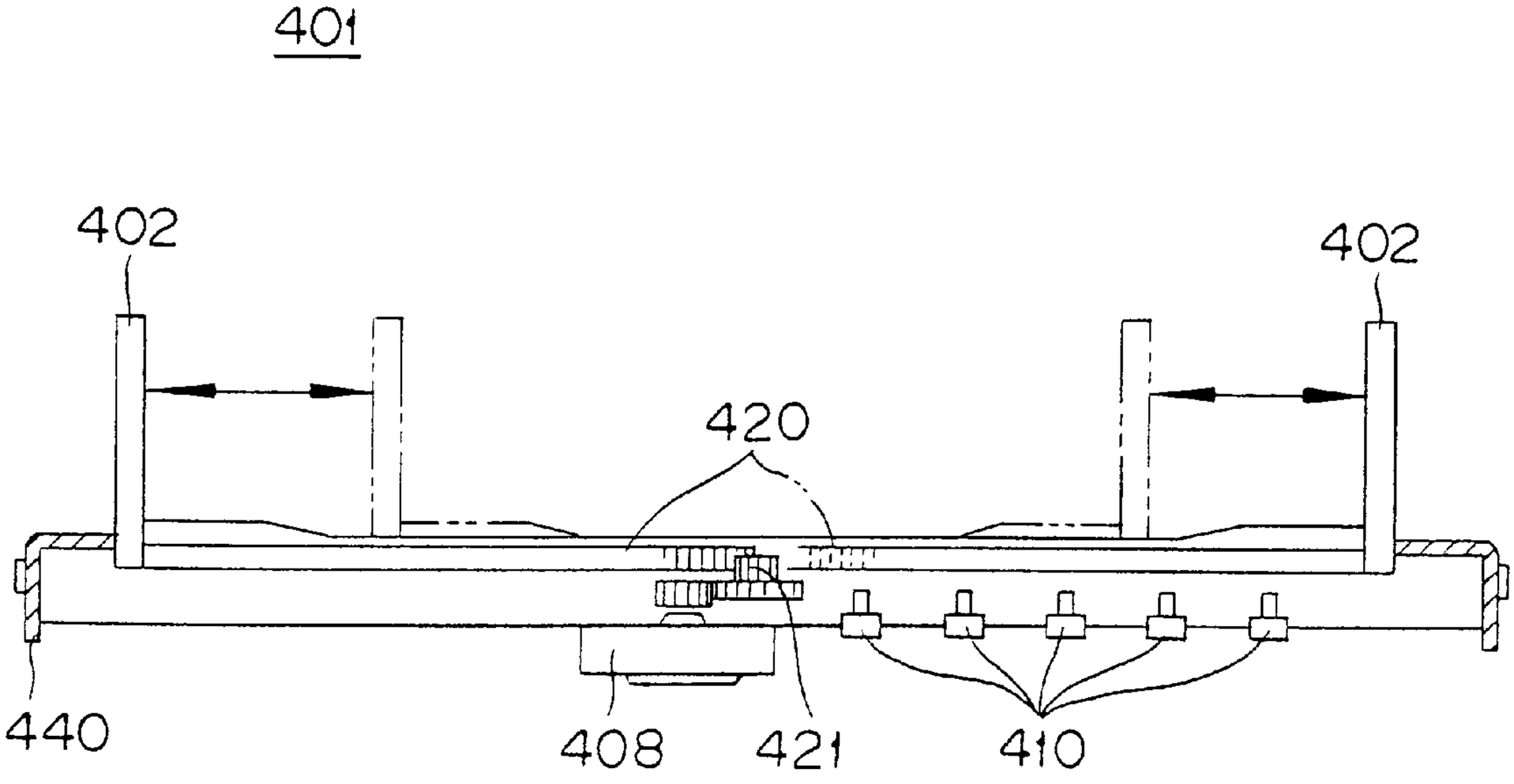


FIG. 17

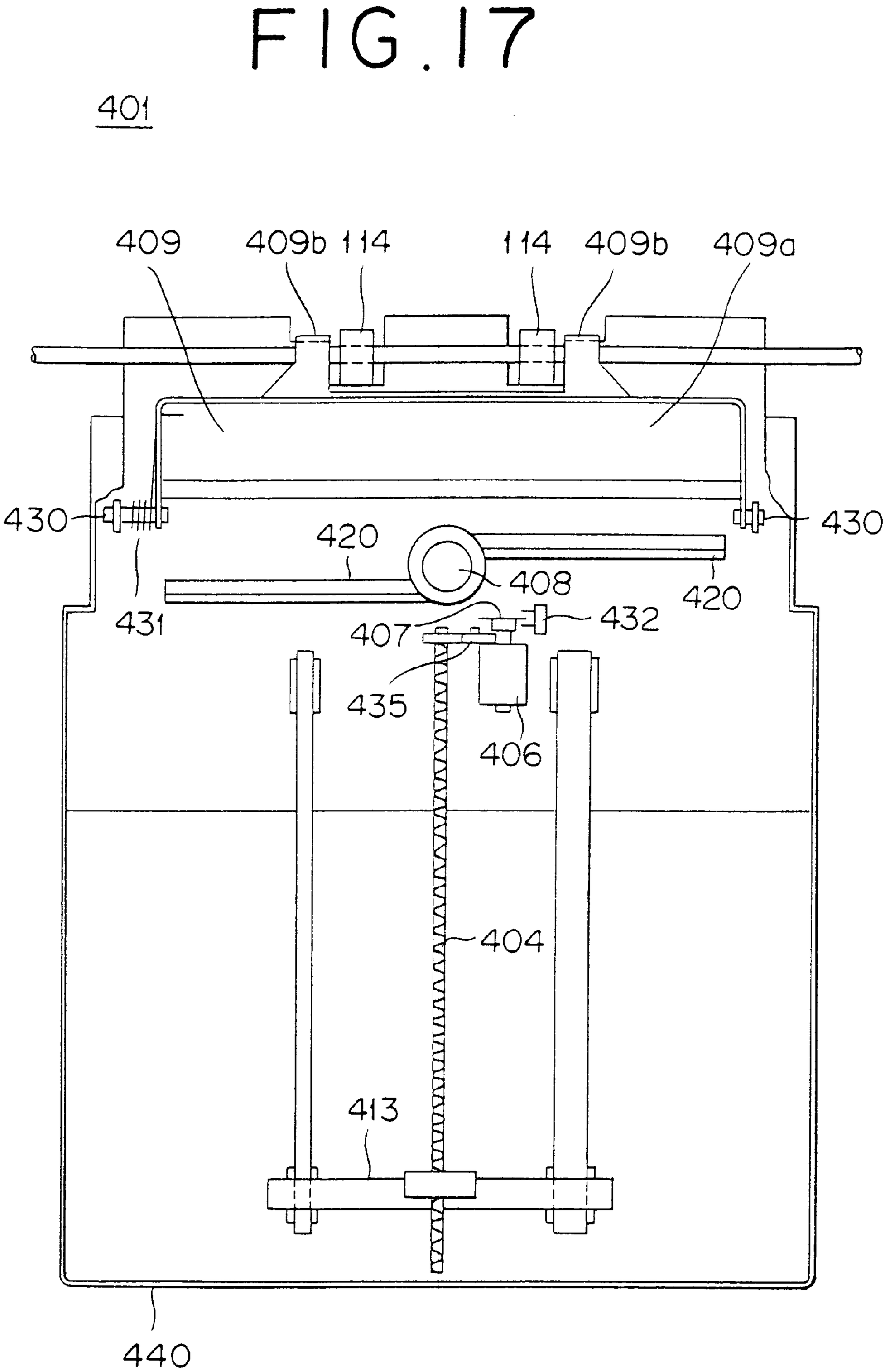

401
409
409b
114
114
409b
409a
430
430
420
408
420
431
407
432
435
406
404
413
440

FIG.18A

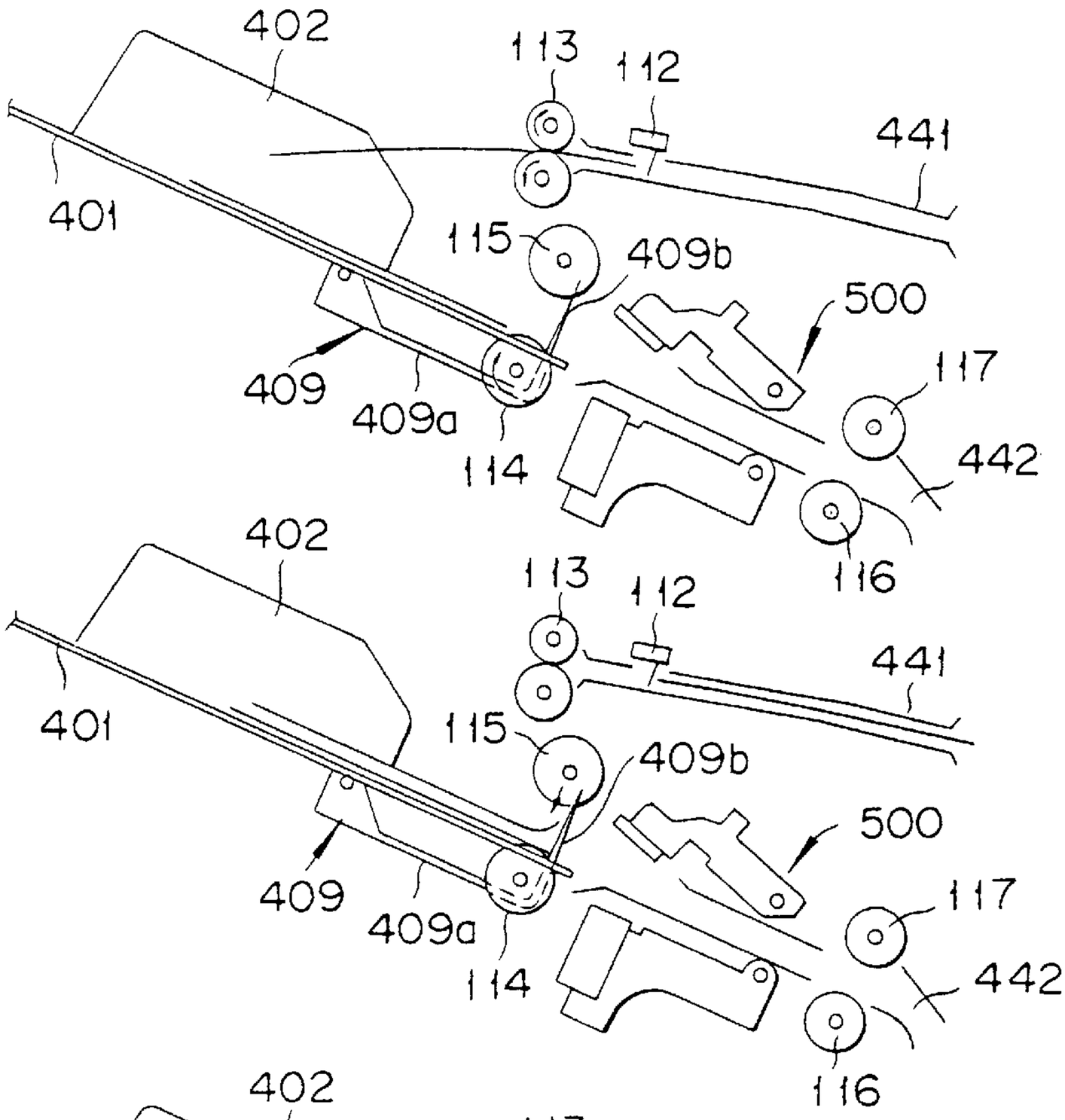


FIG.18B

FIG.18C

FIG.18D

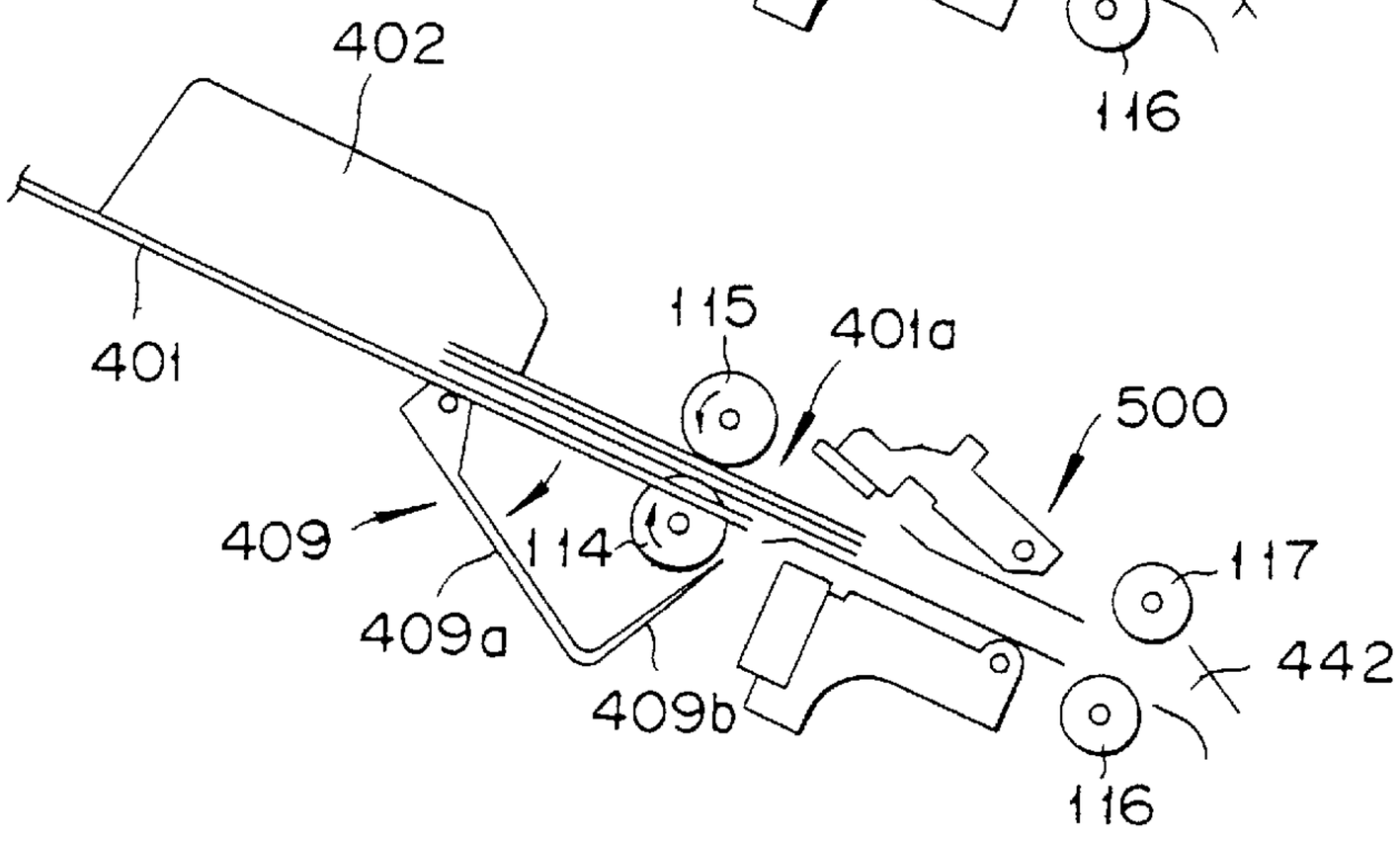


FIG. 19A

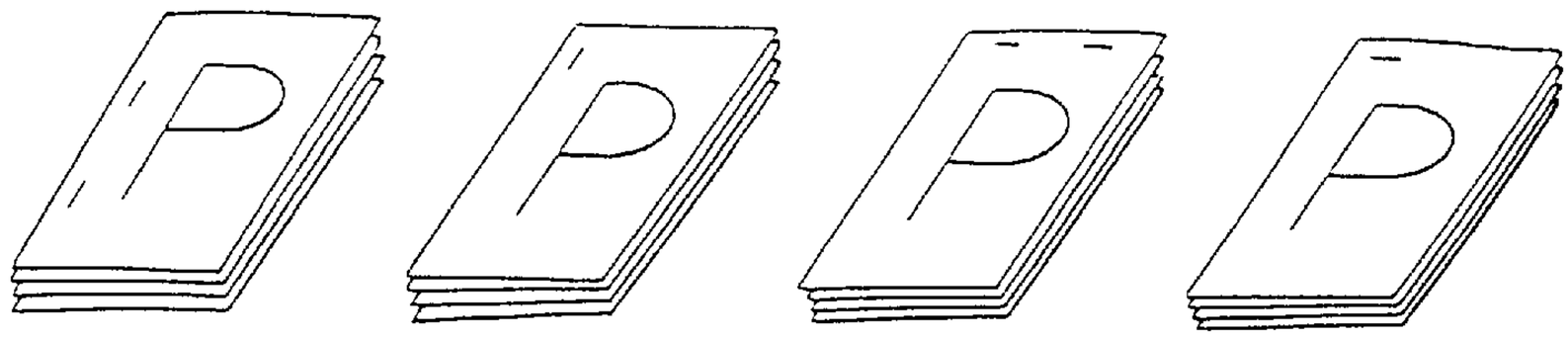
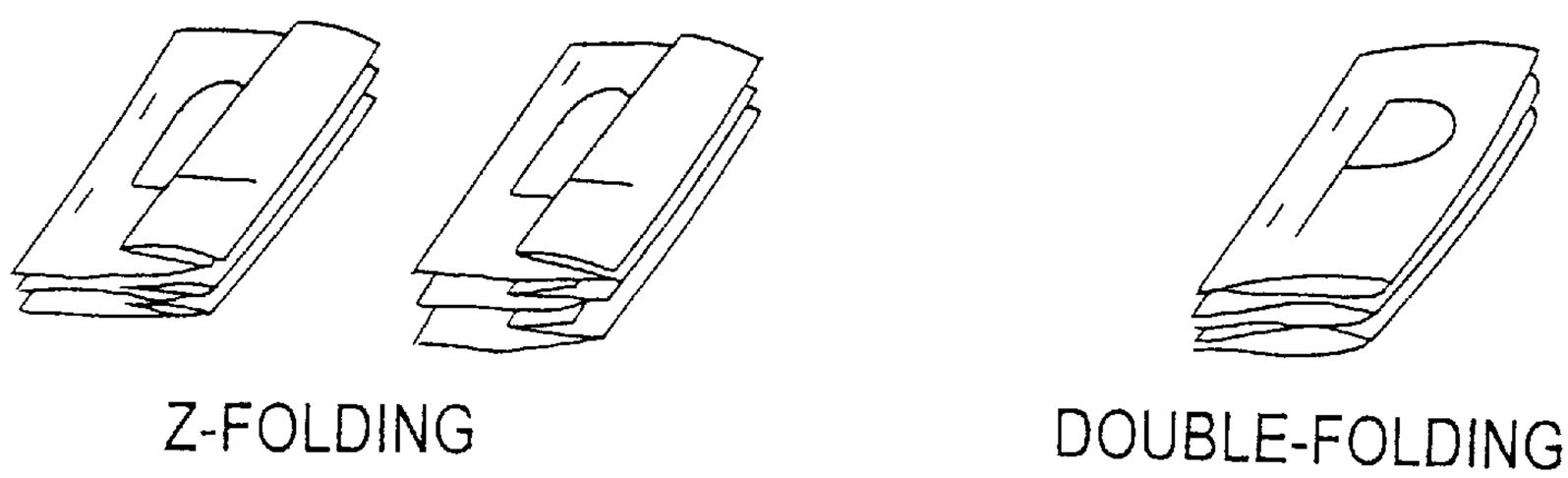


FIG. 19B



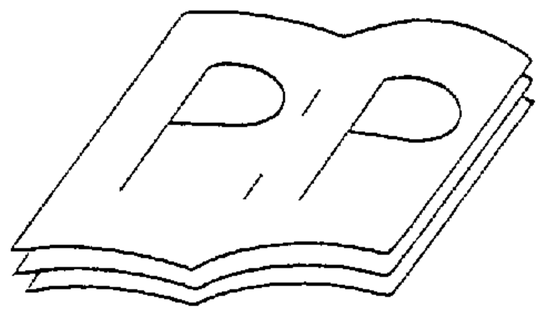


FIG. 19C

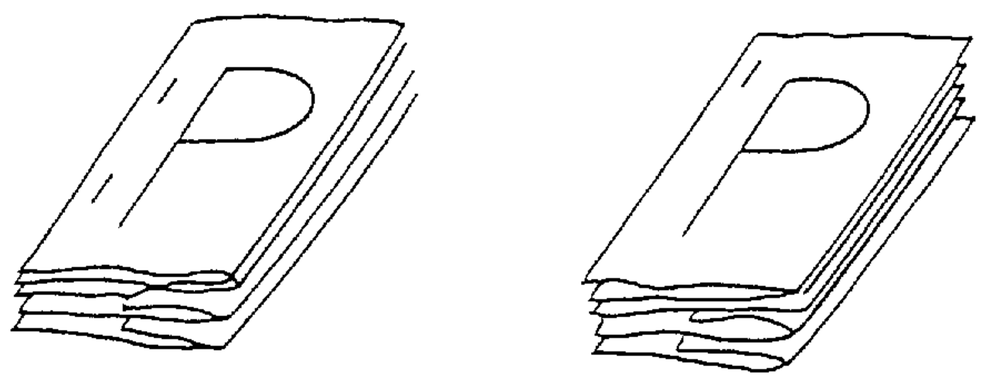


FIG.20

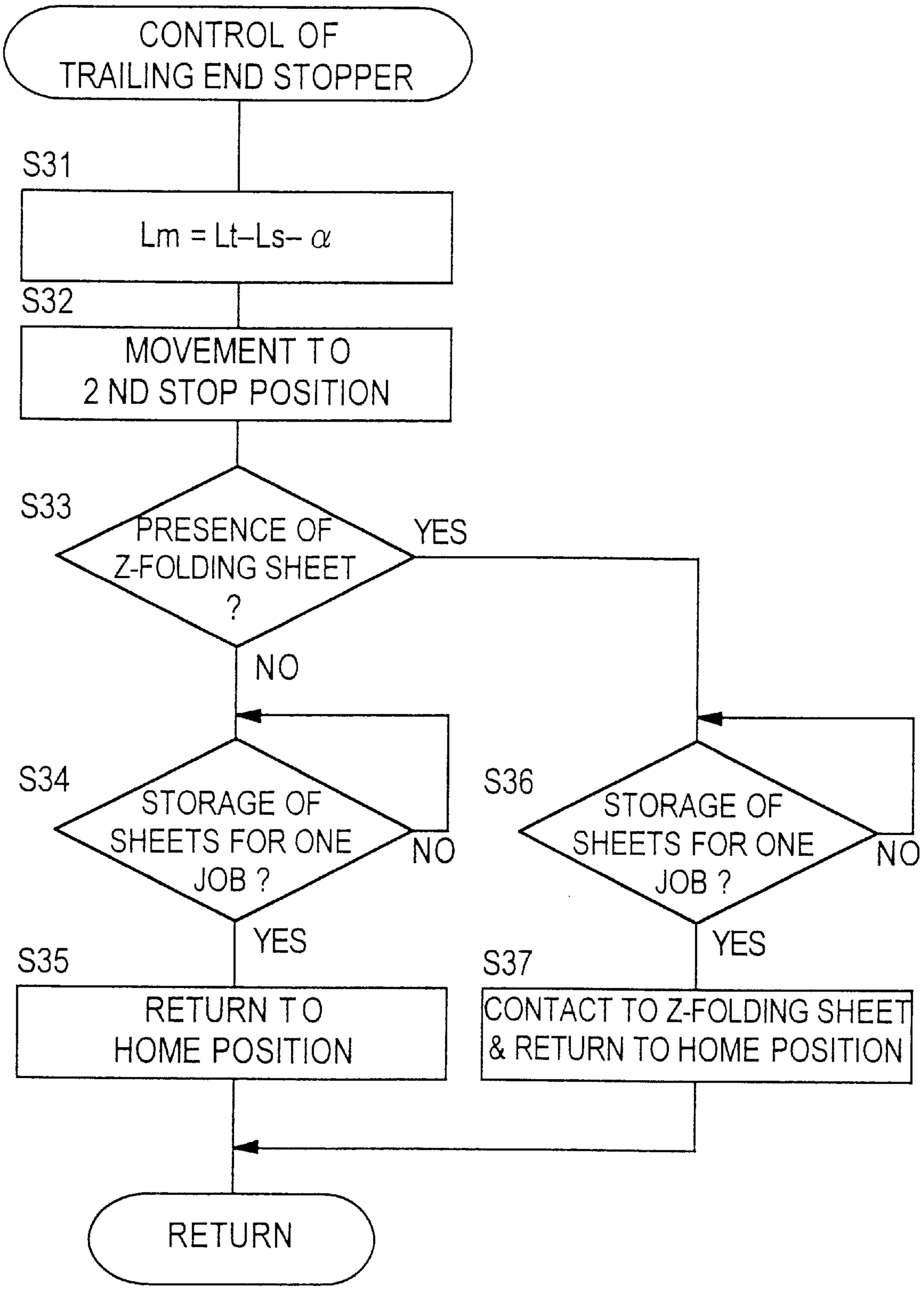

CONTROL OF TRAILING END STOPPER
S31
Lm = Lt–Ls– α
S32
MOVEMENT TO 2 ND STOP POSITION
S33
PRESENCE OF Z-FOLDING SHEET ?
YES
NO
S34
STORAGE OF SHEETS FOR ONE JOB ?
NO
YES
S35
RETURN TO HOME POSITION
S36
STORAGE OF SHEETS FOR ONE JOB ?
NO
YES
S37
CONTACT TO Z-FOLDING SHEET & RETURN TO HOME POSITION
RETURN

FIG.21

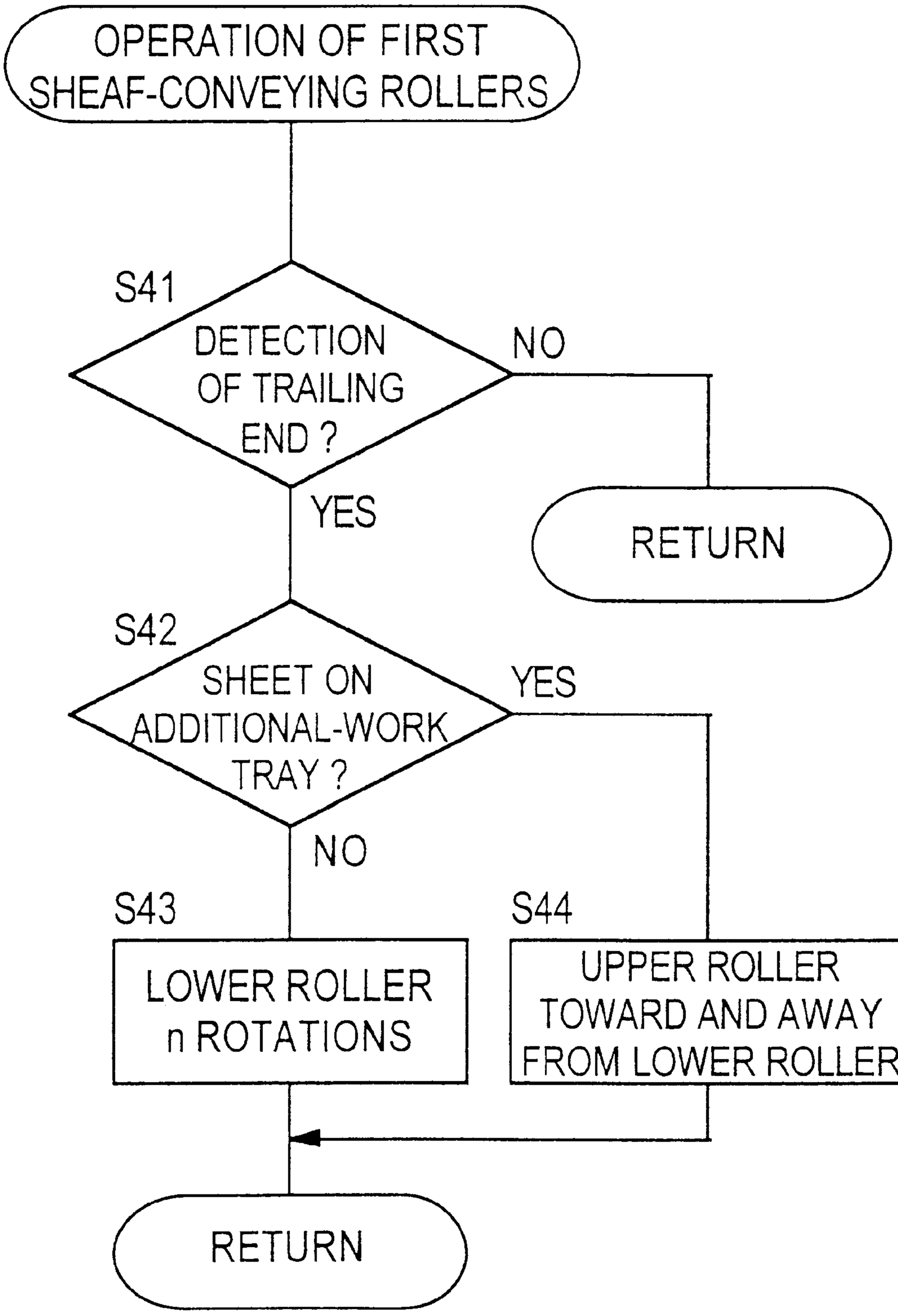

OPERATION OF FIRST SHEAF-CONVEYING ROLLERS
S41
DETECTION OF TRAILING END ?
NO
YES
RETURN
S42
SHEET ON ADDITIONAL-WORK TRAY ?
YES
NO
S43
LOWER ROLLER n ROTATIONS
S44
UPPER ROLLER TOWARD AND AWAY FROM LOWER ROLLER
RETURN

FIG. 22A

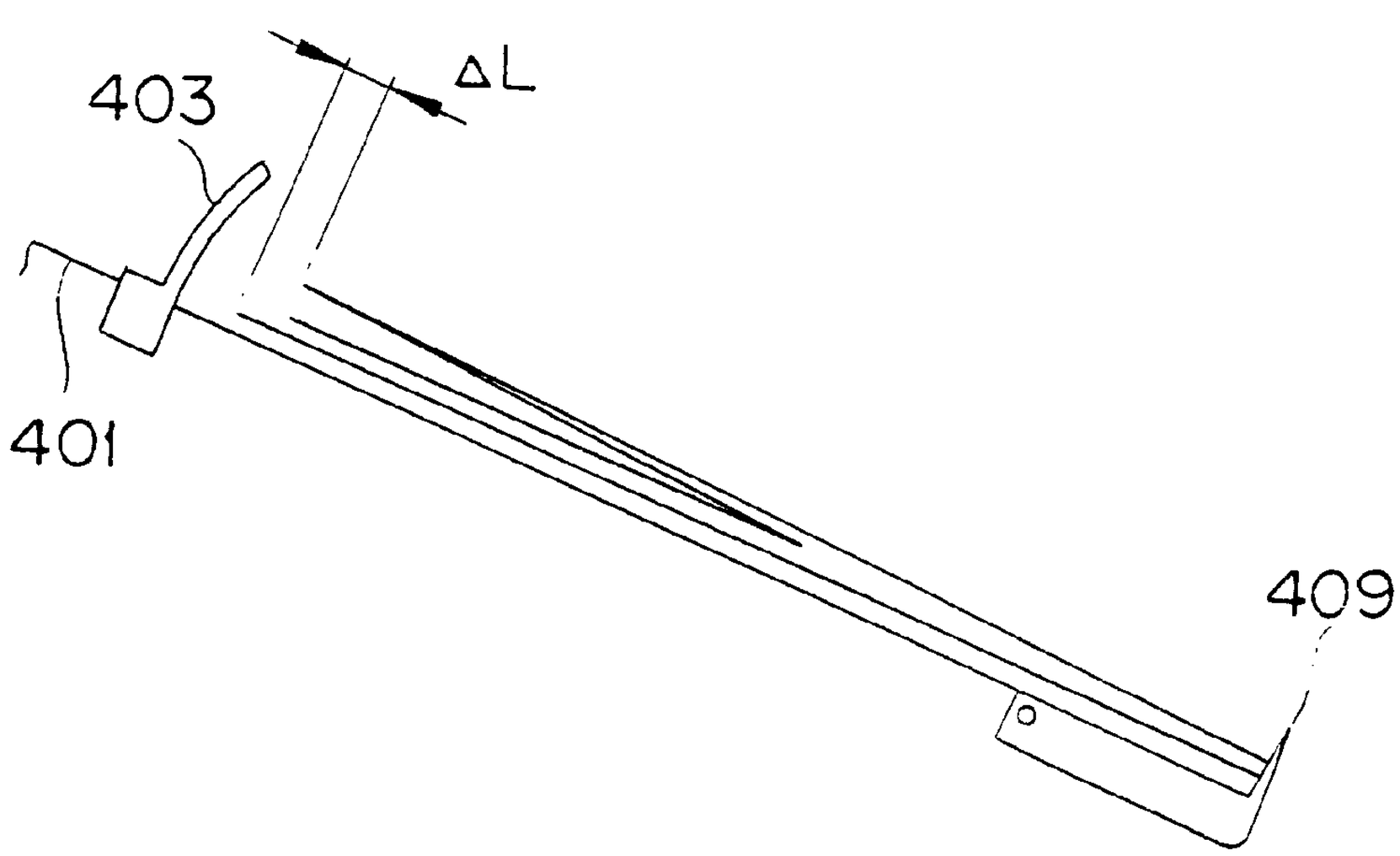


FIG. 22B

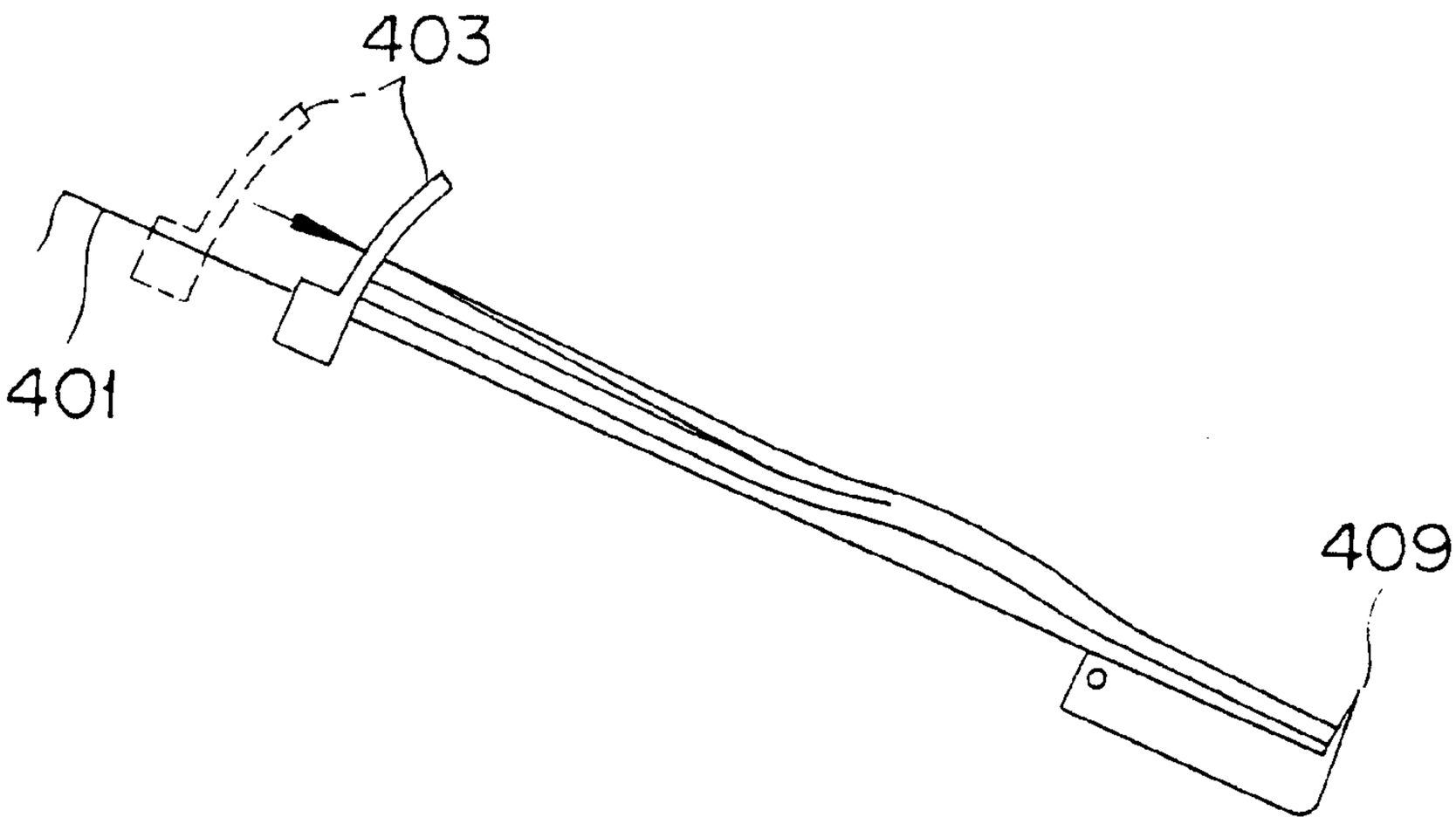
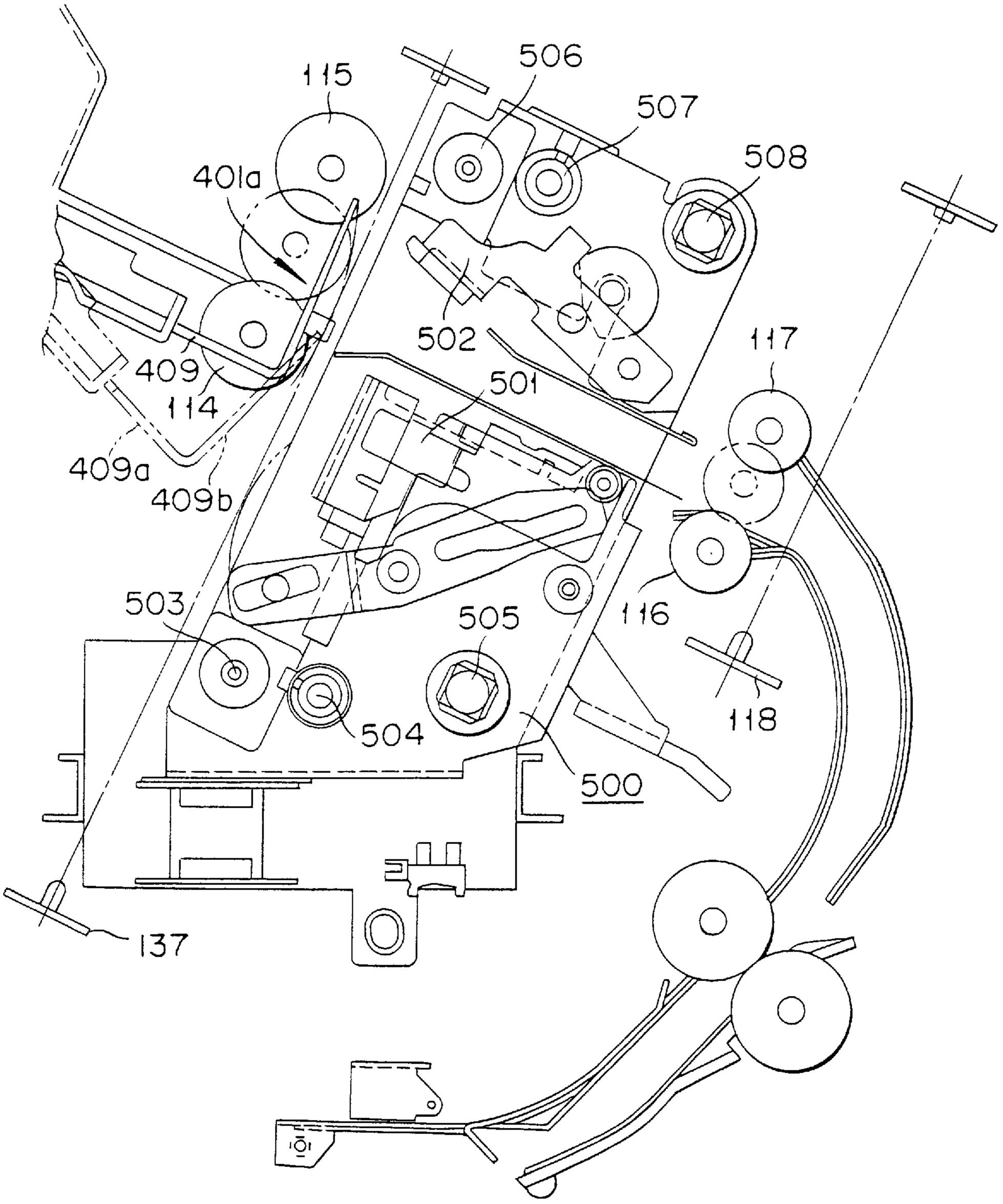


FIG. 23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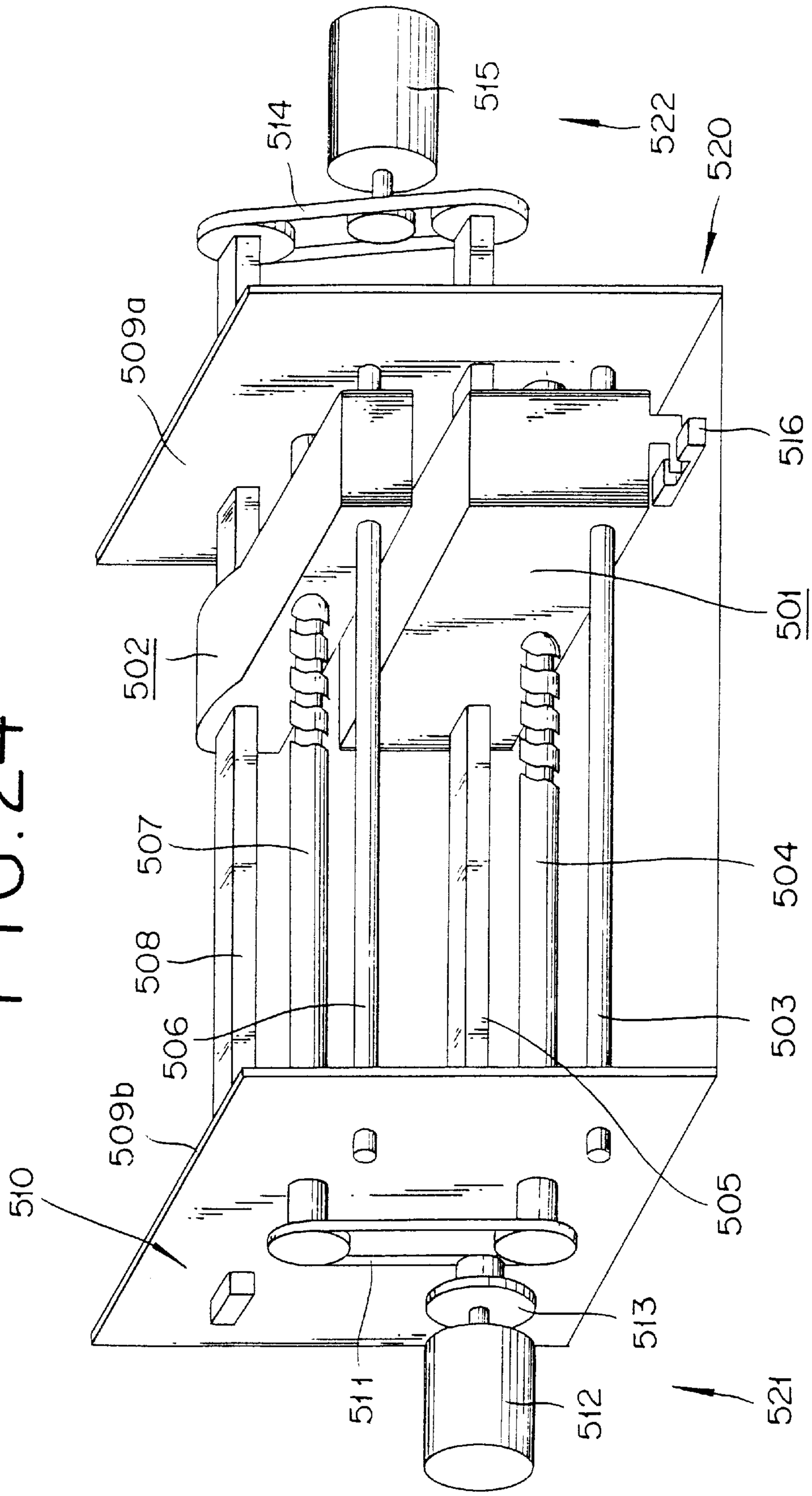
500
137


514
515
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509a
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503
509b
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513
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512
521
500

FIG. 24

FIG. 25A

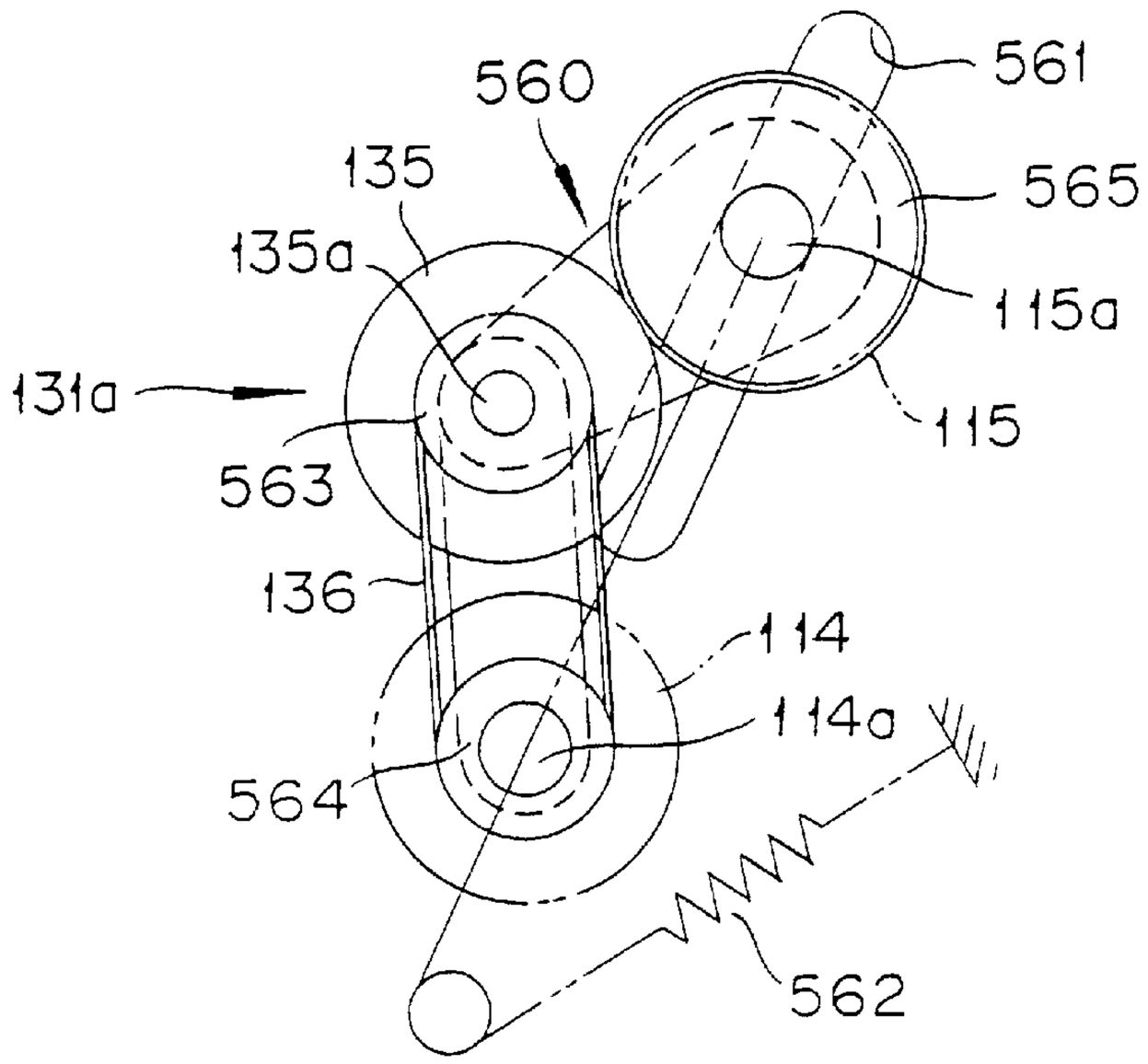


FIG. 25B

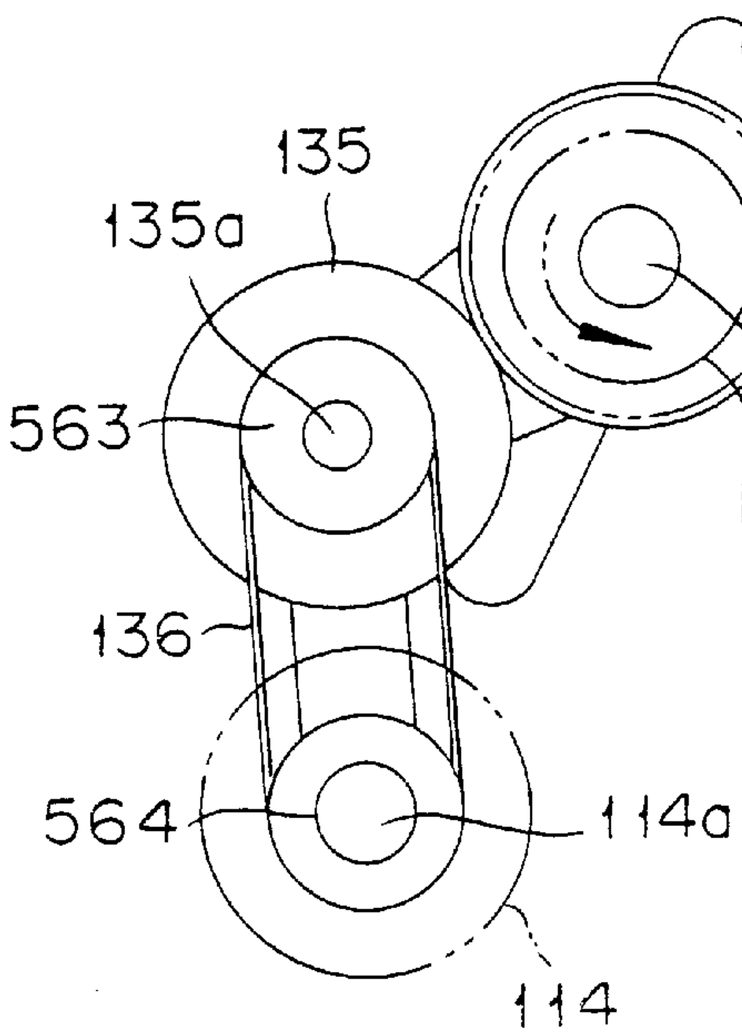


FIG. 25C

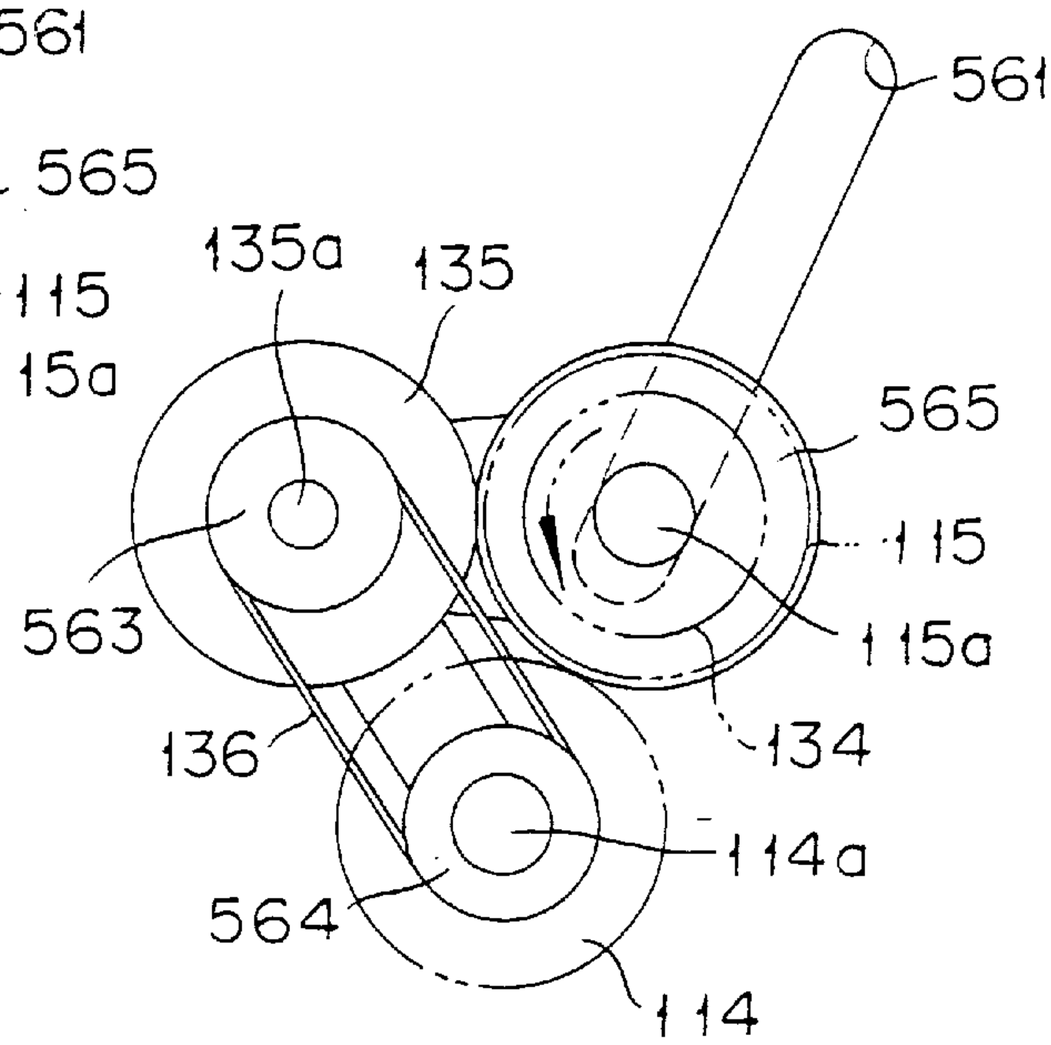


FIG. 26

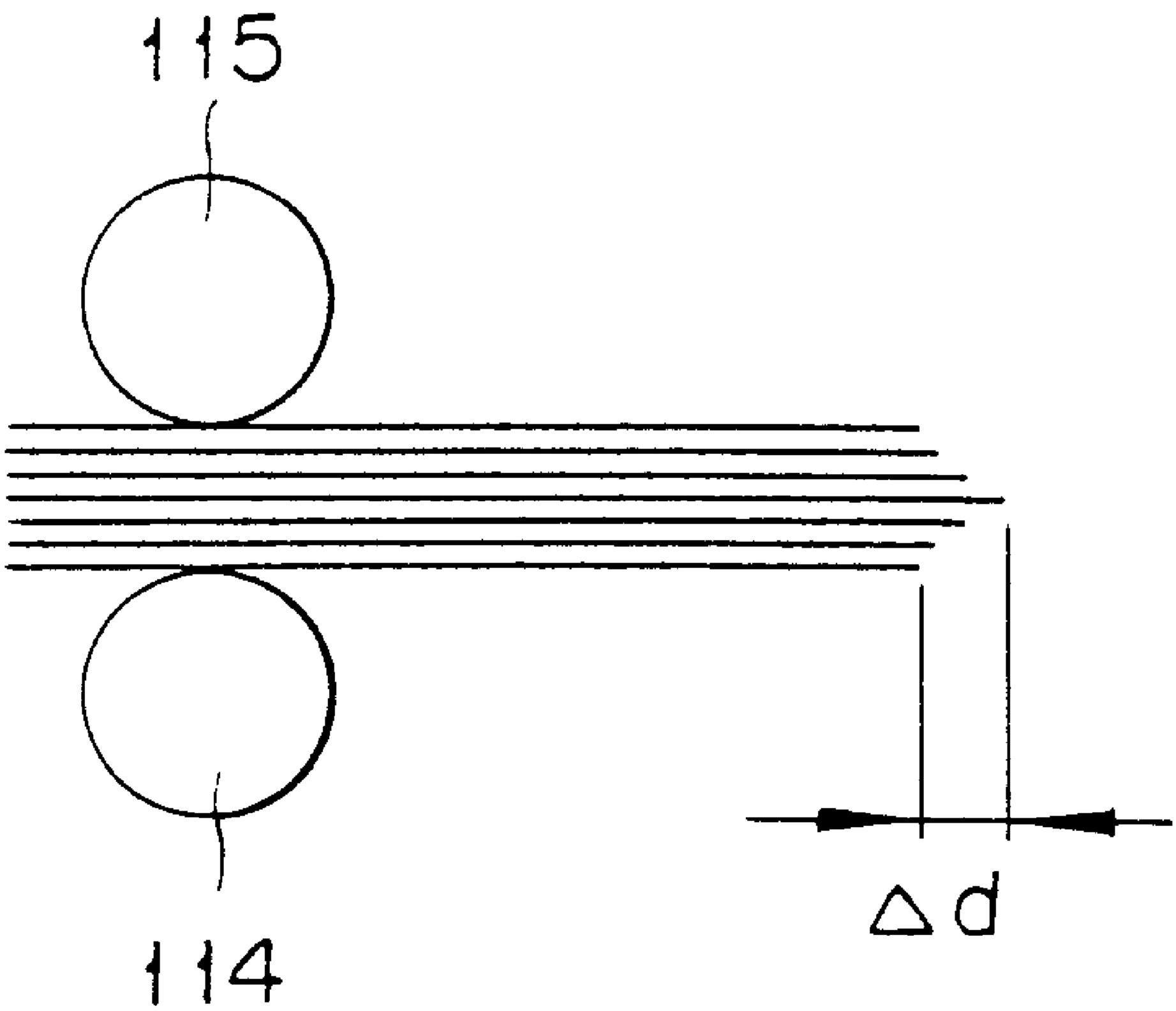


FIG. 27A

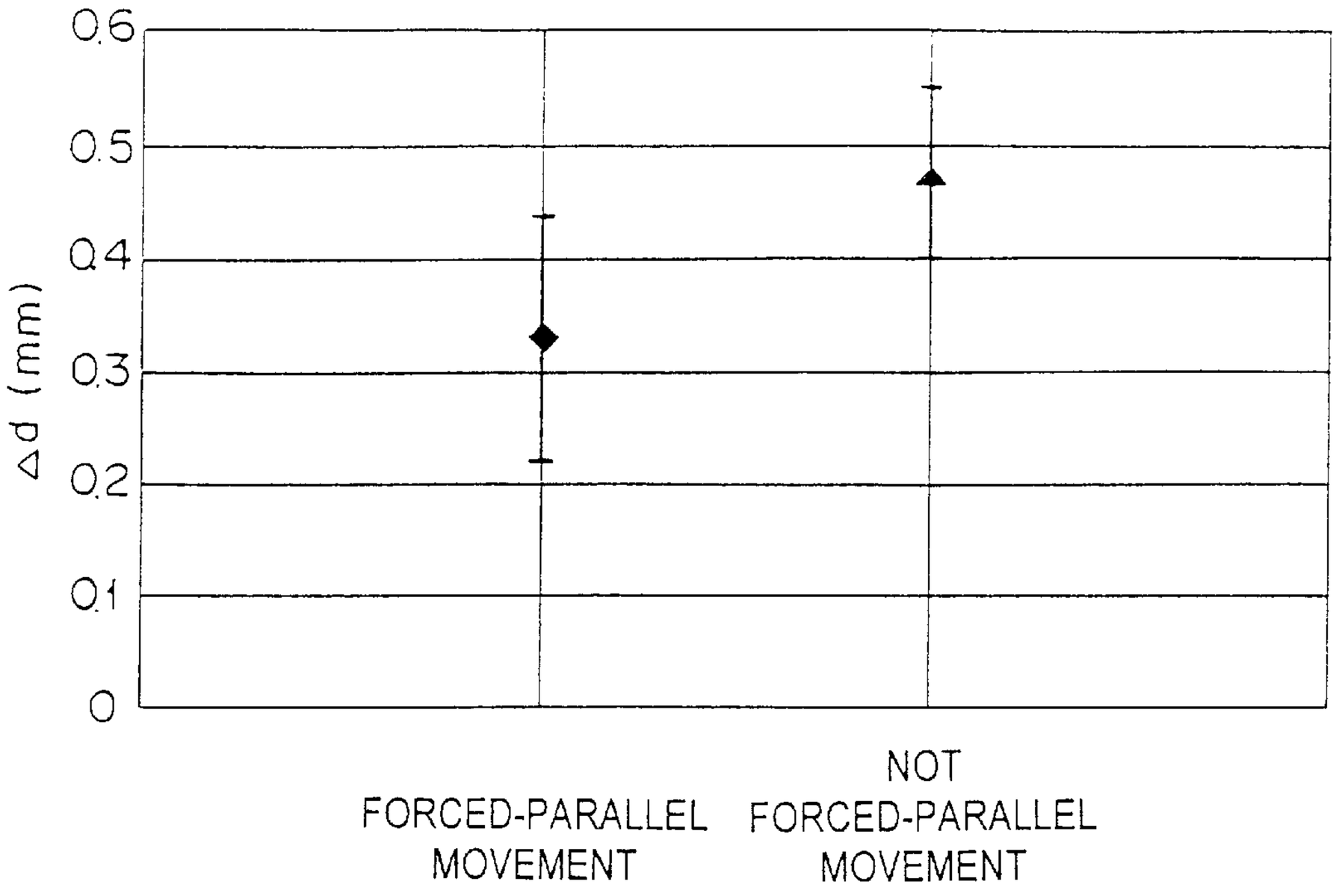


FIG. 27B

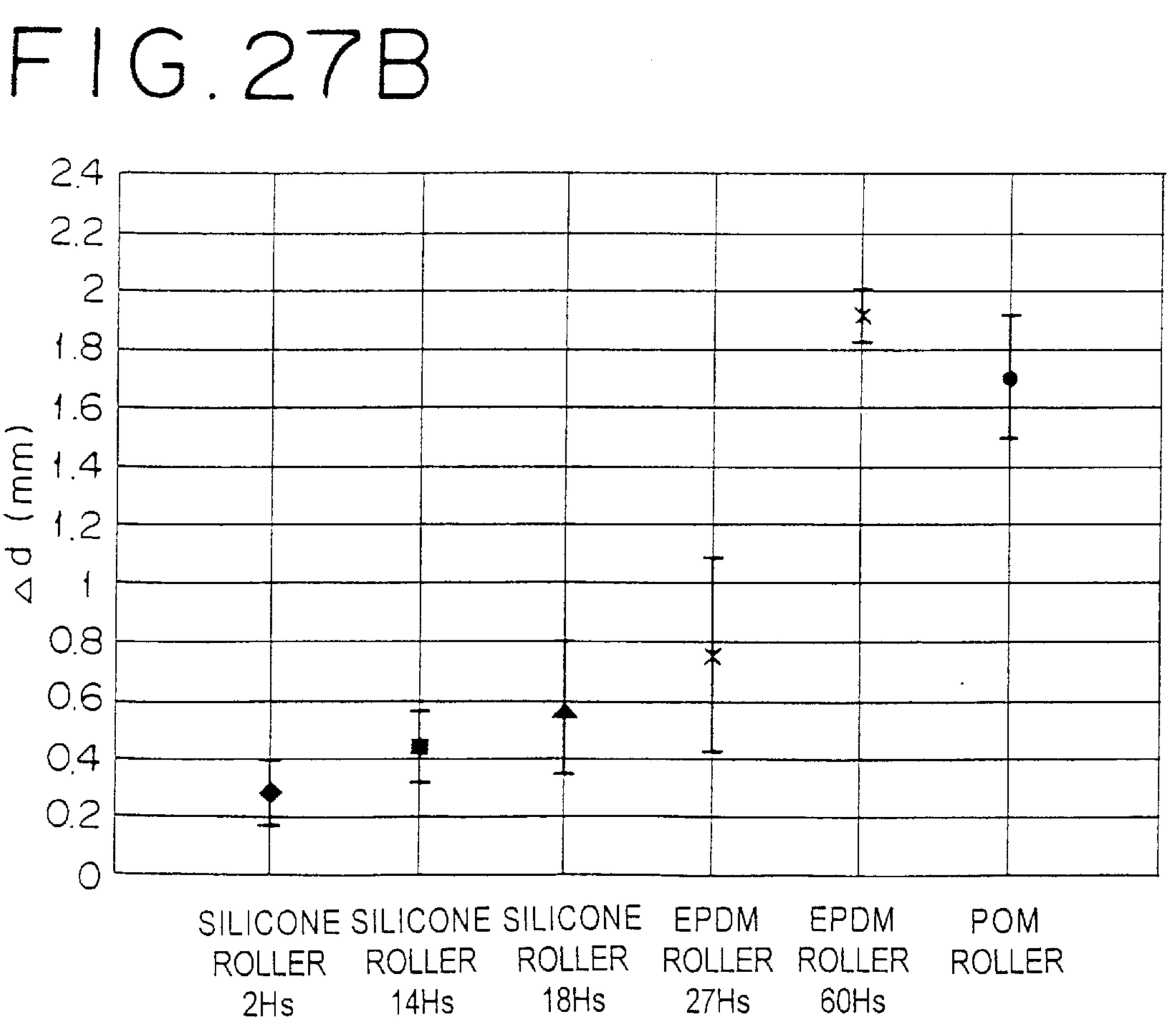


FIG. 28A

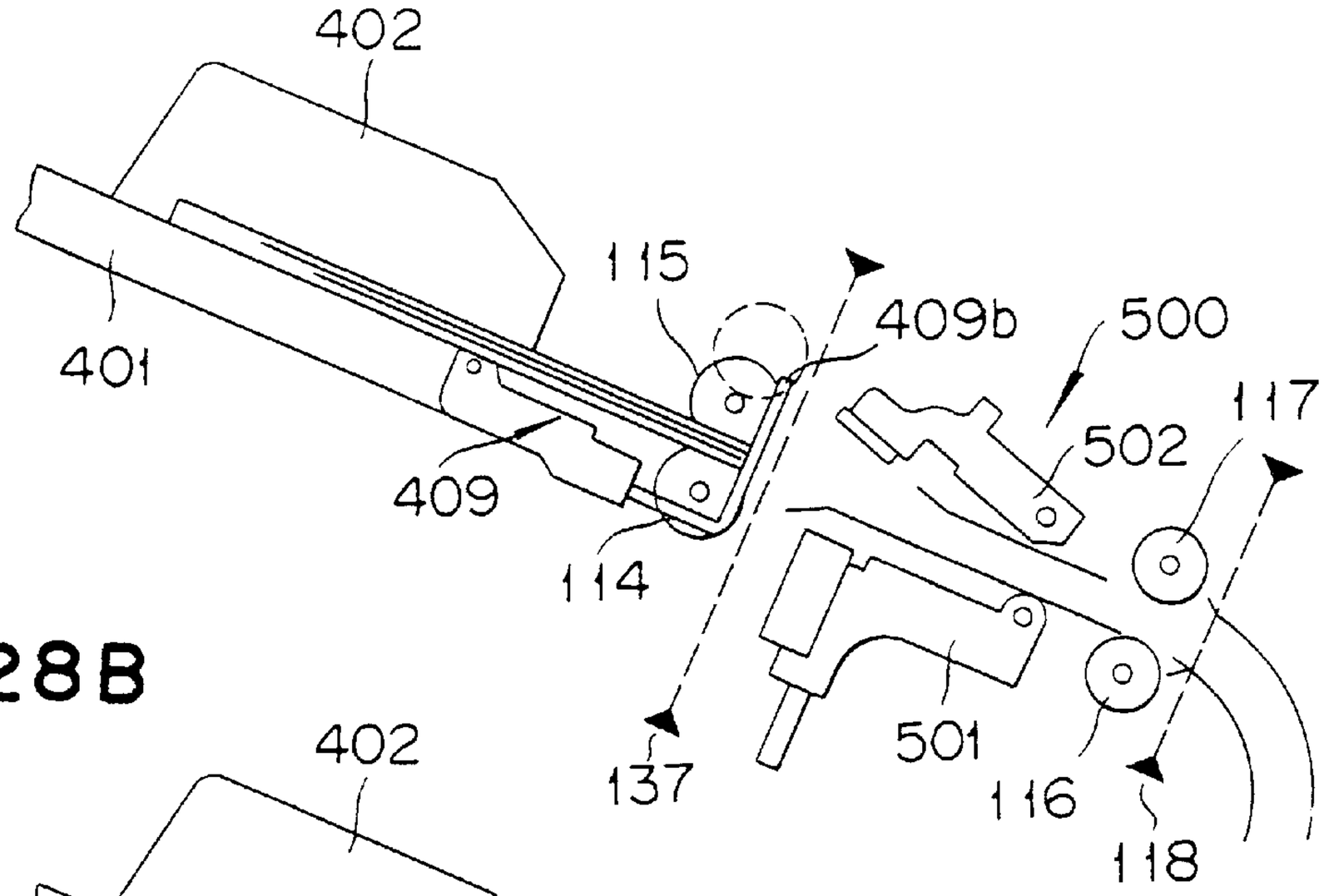


FIG. 28B

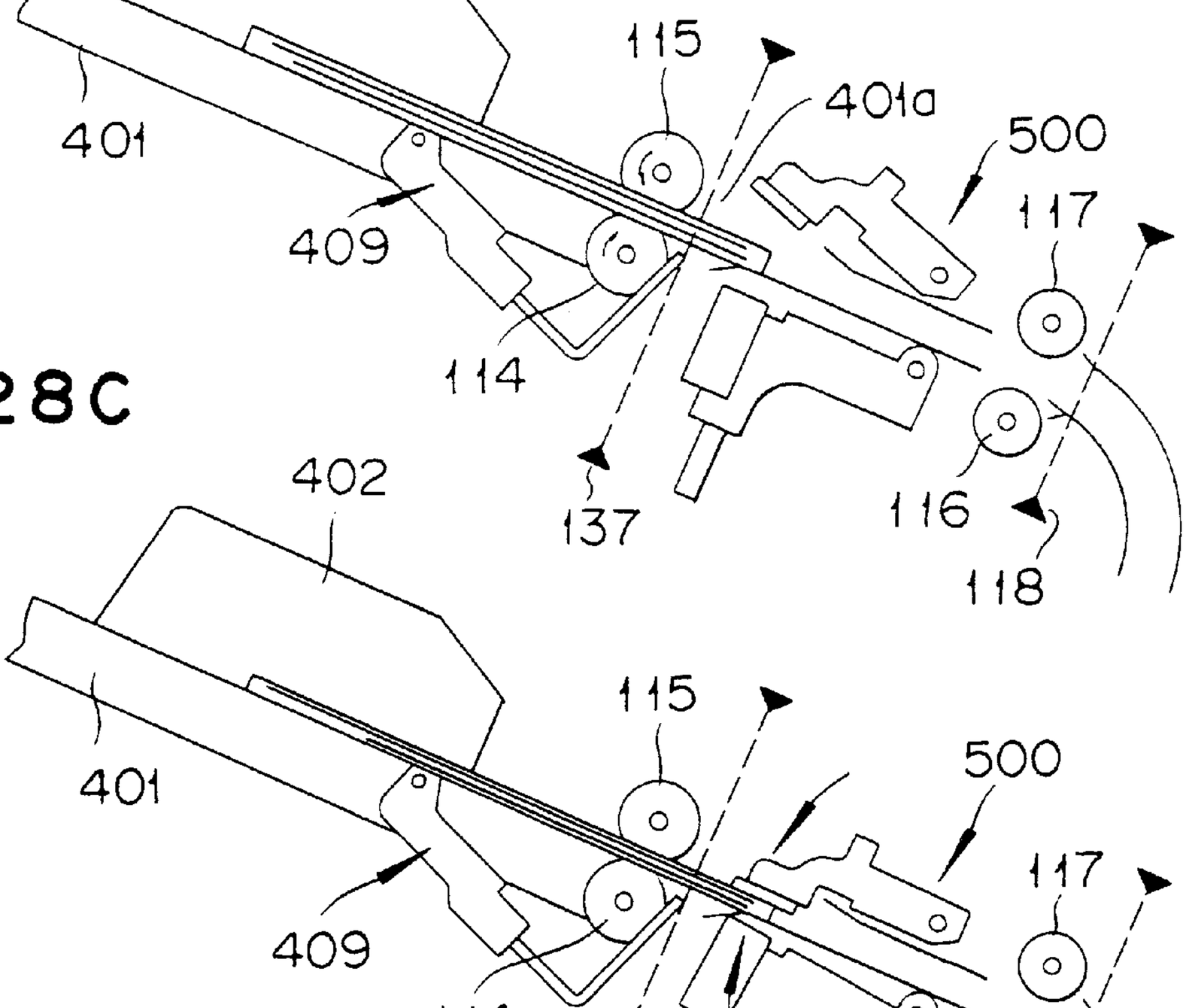


FIG. 28C

FIG. 28D

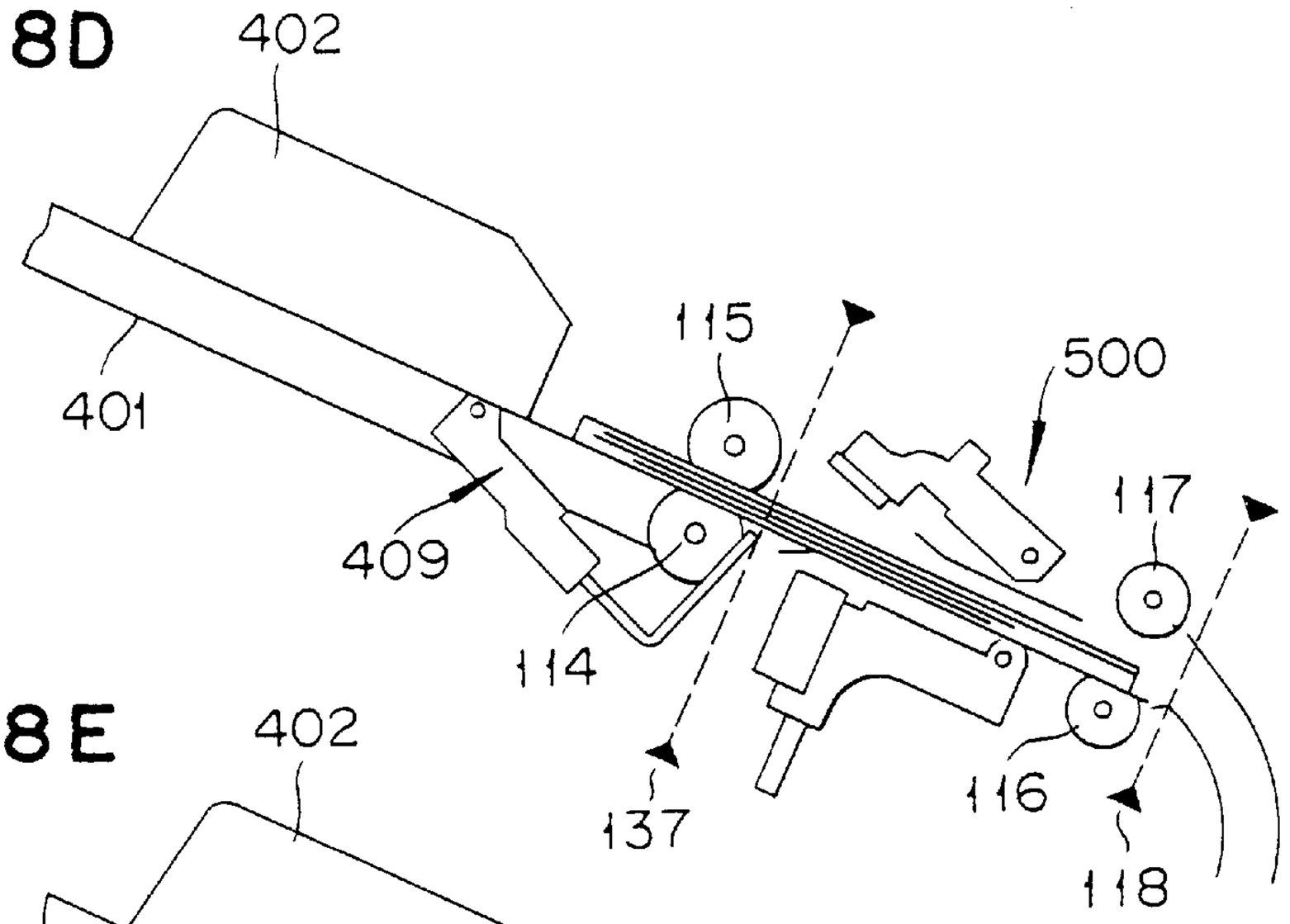


FIG. 28E

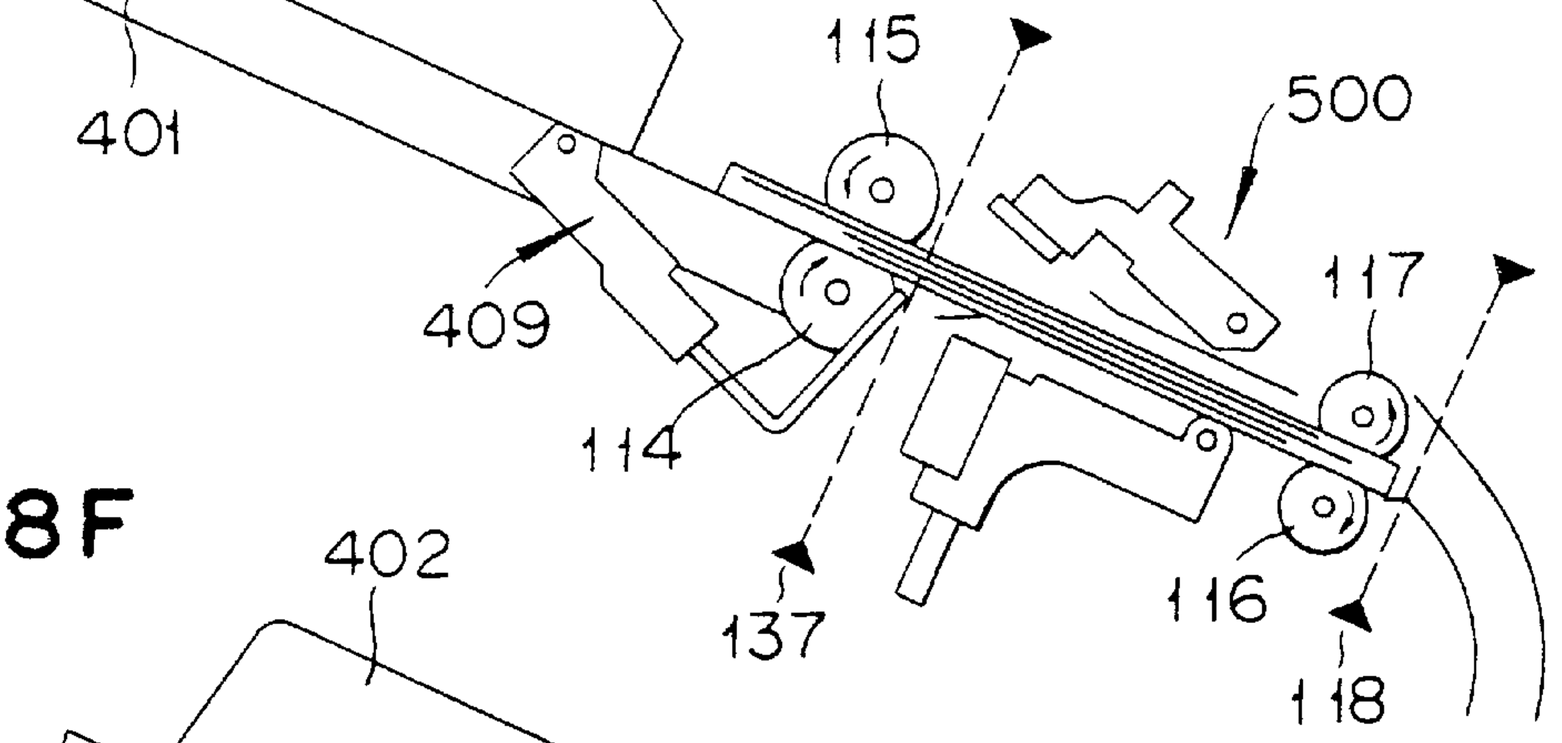


FIG. 28F

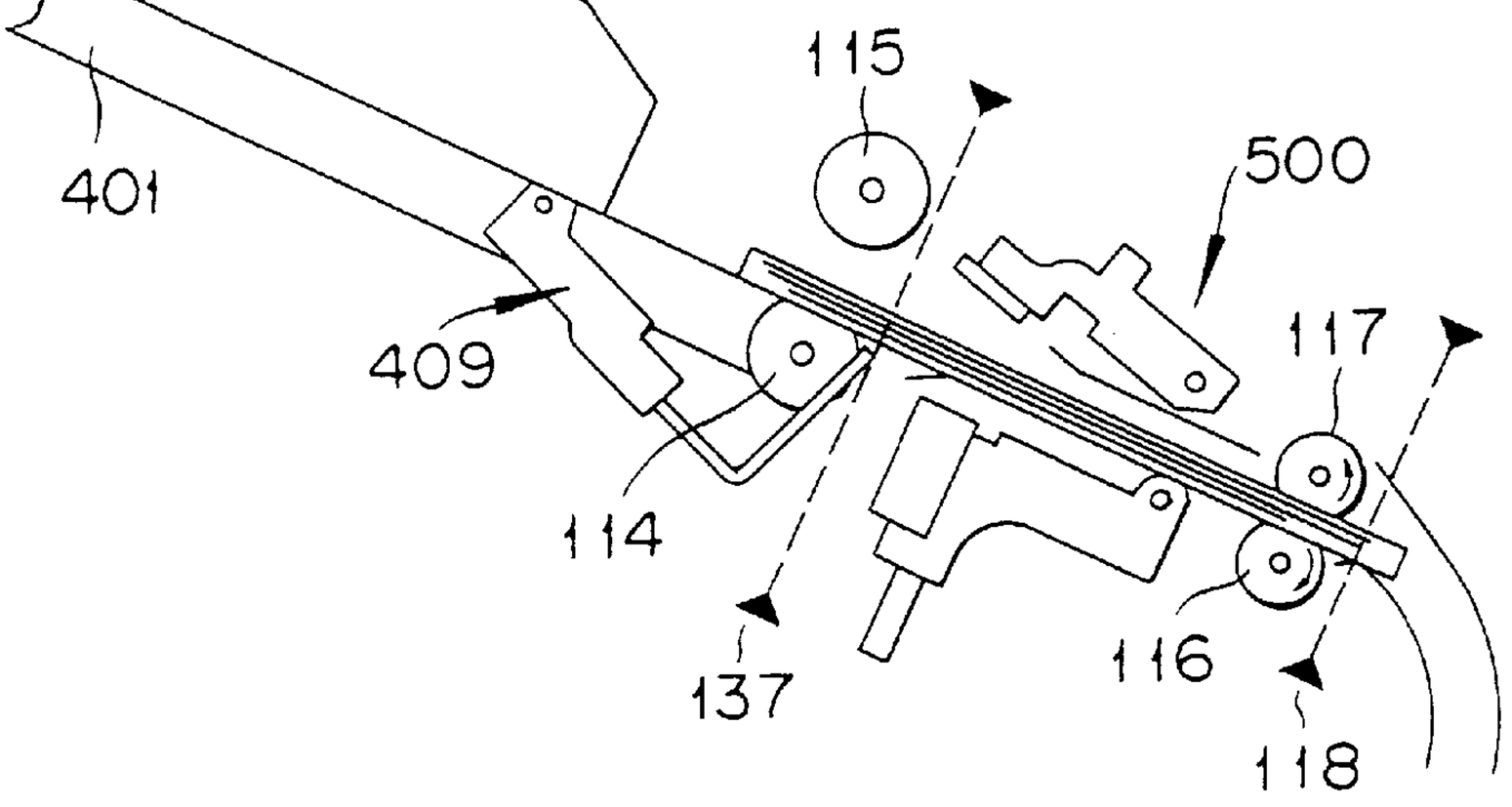


FIG. 29A

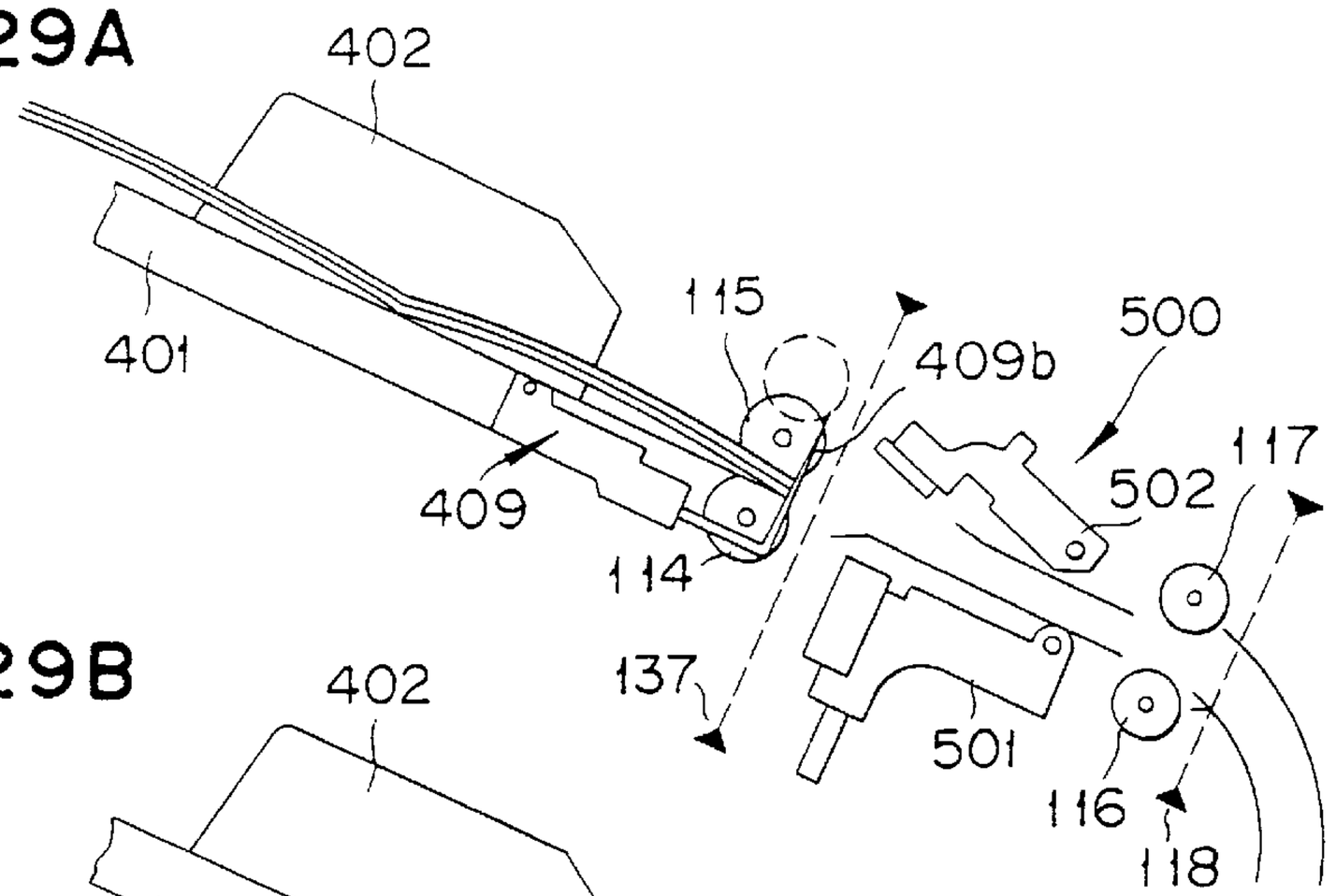


FIG. 29B

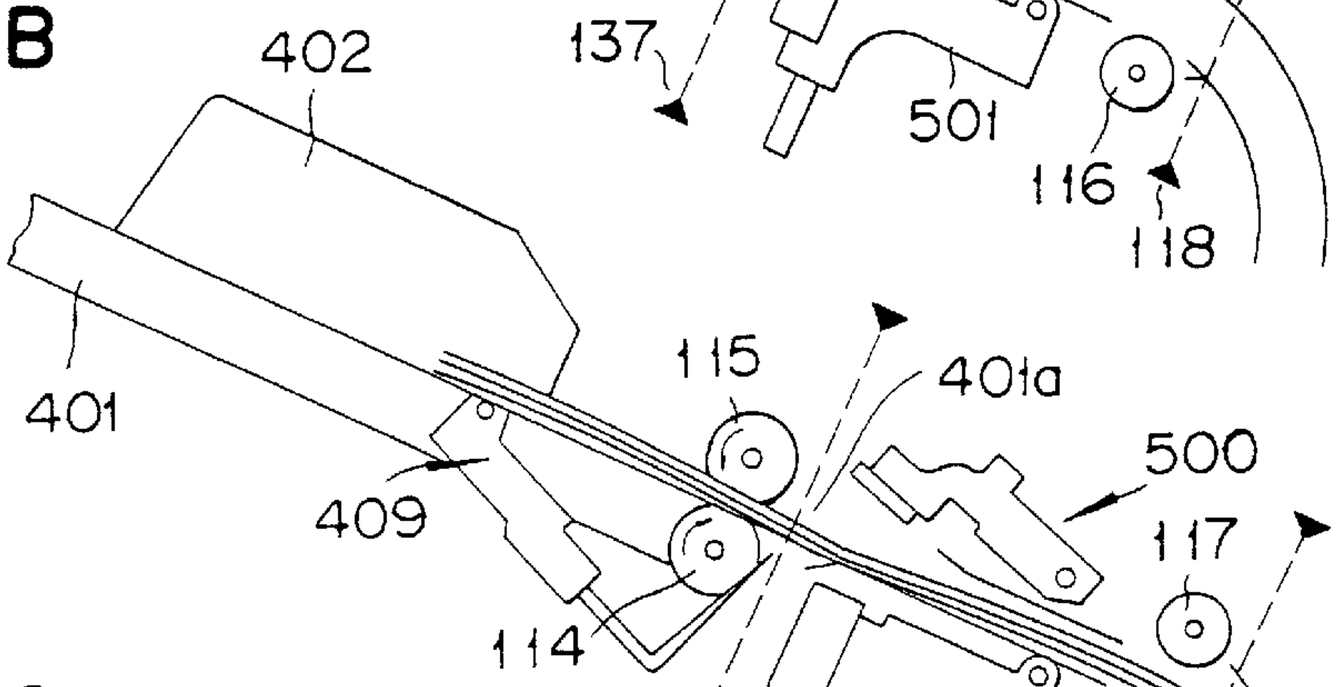


FIG. 29C

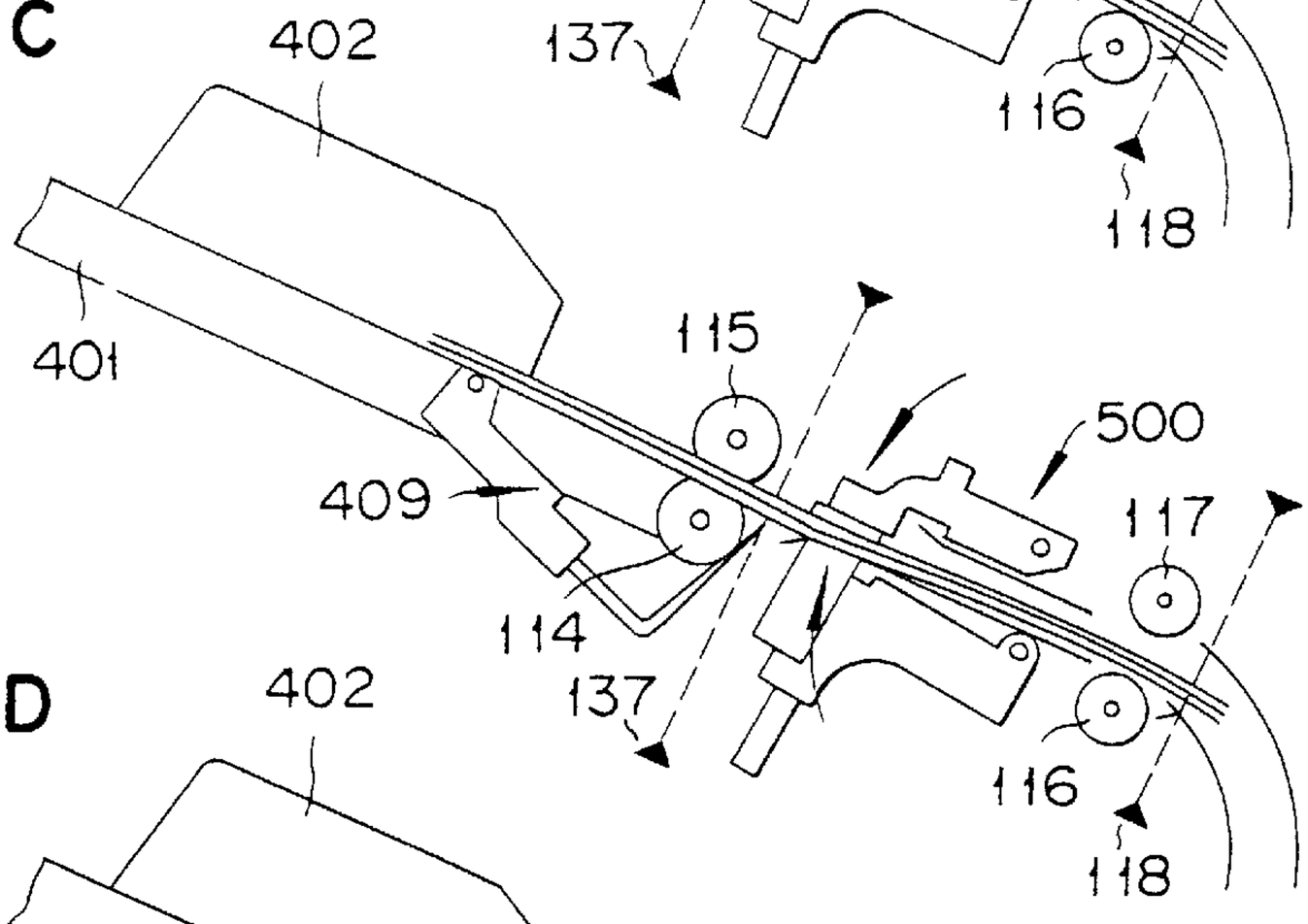


FIG. 29D

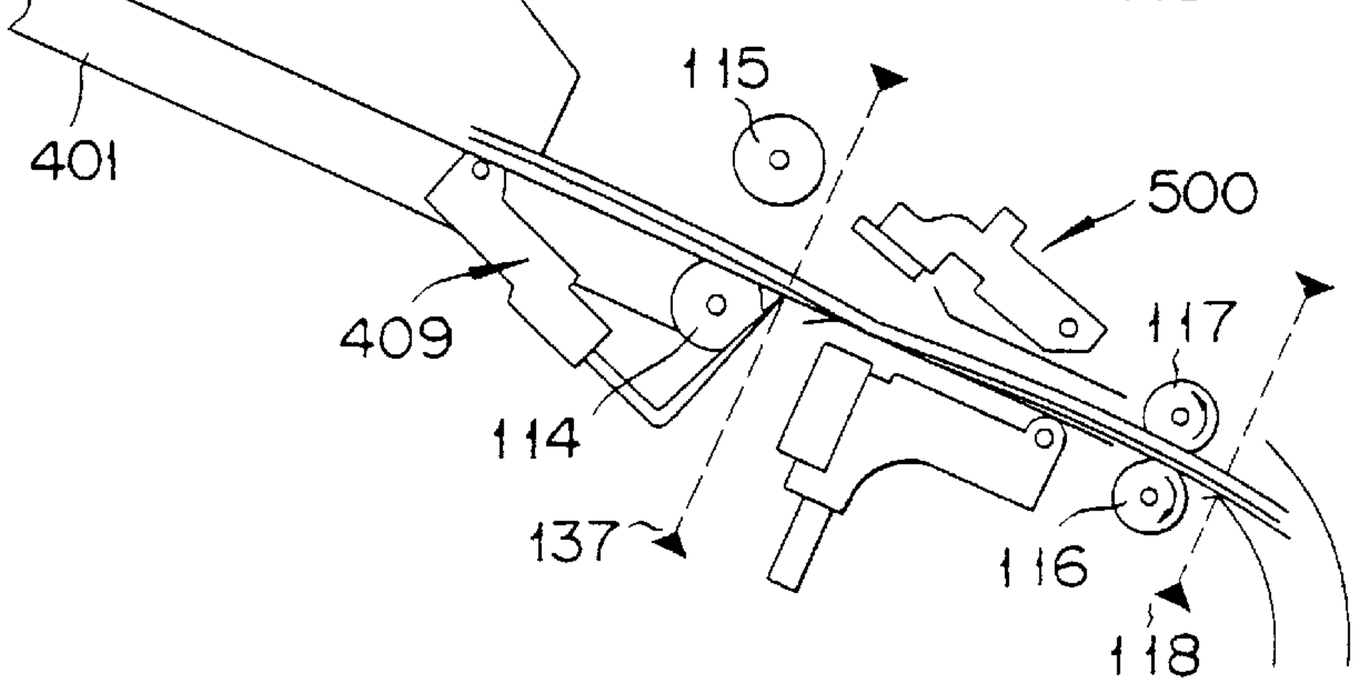


FIG. 30A

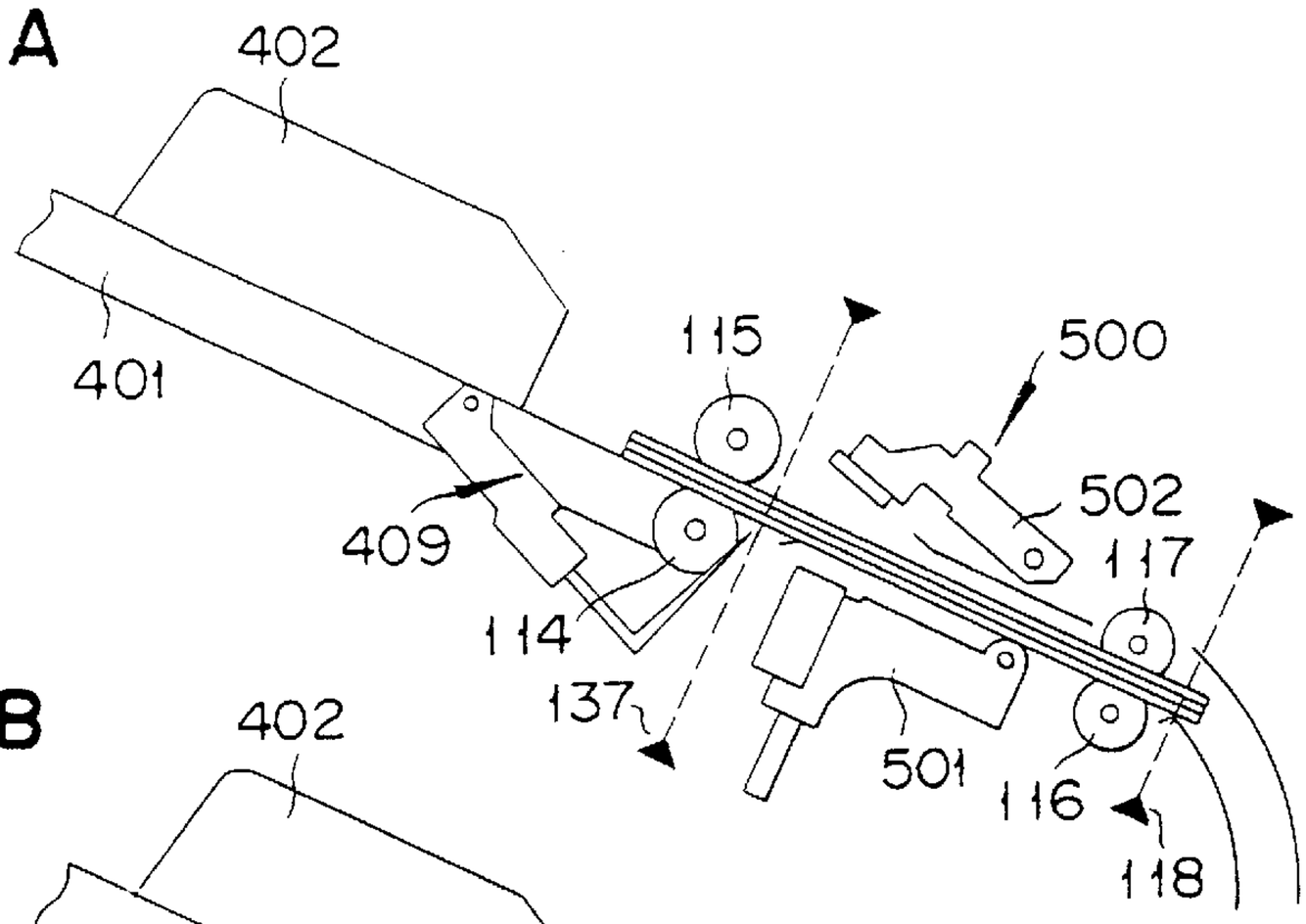


FIG. 30B

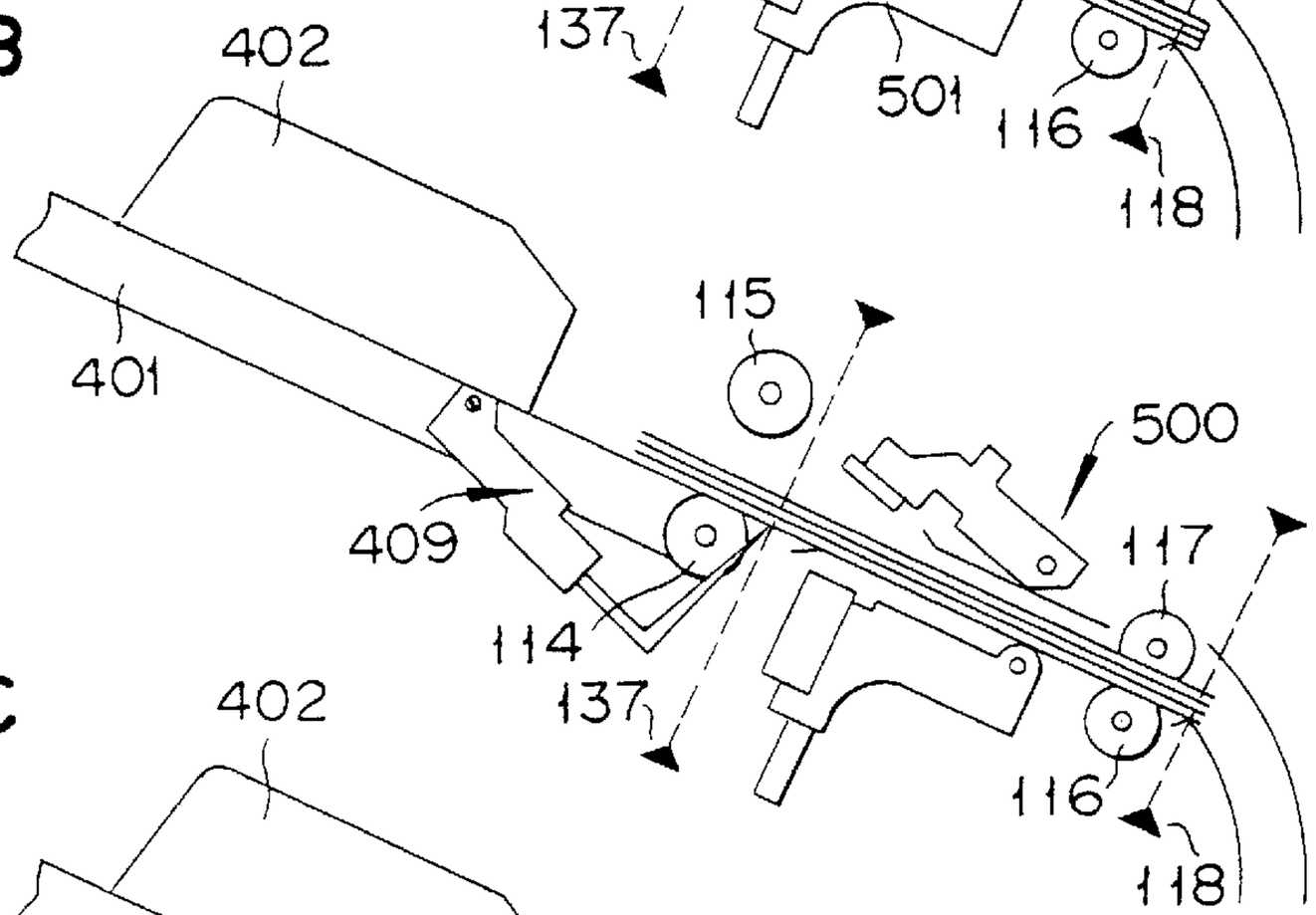


FIG. 30C

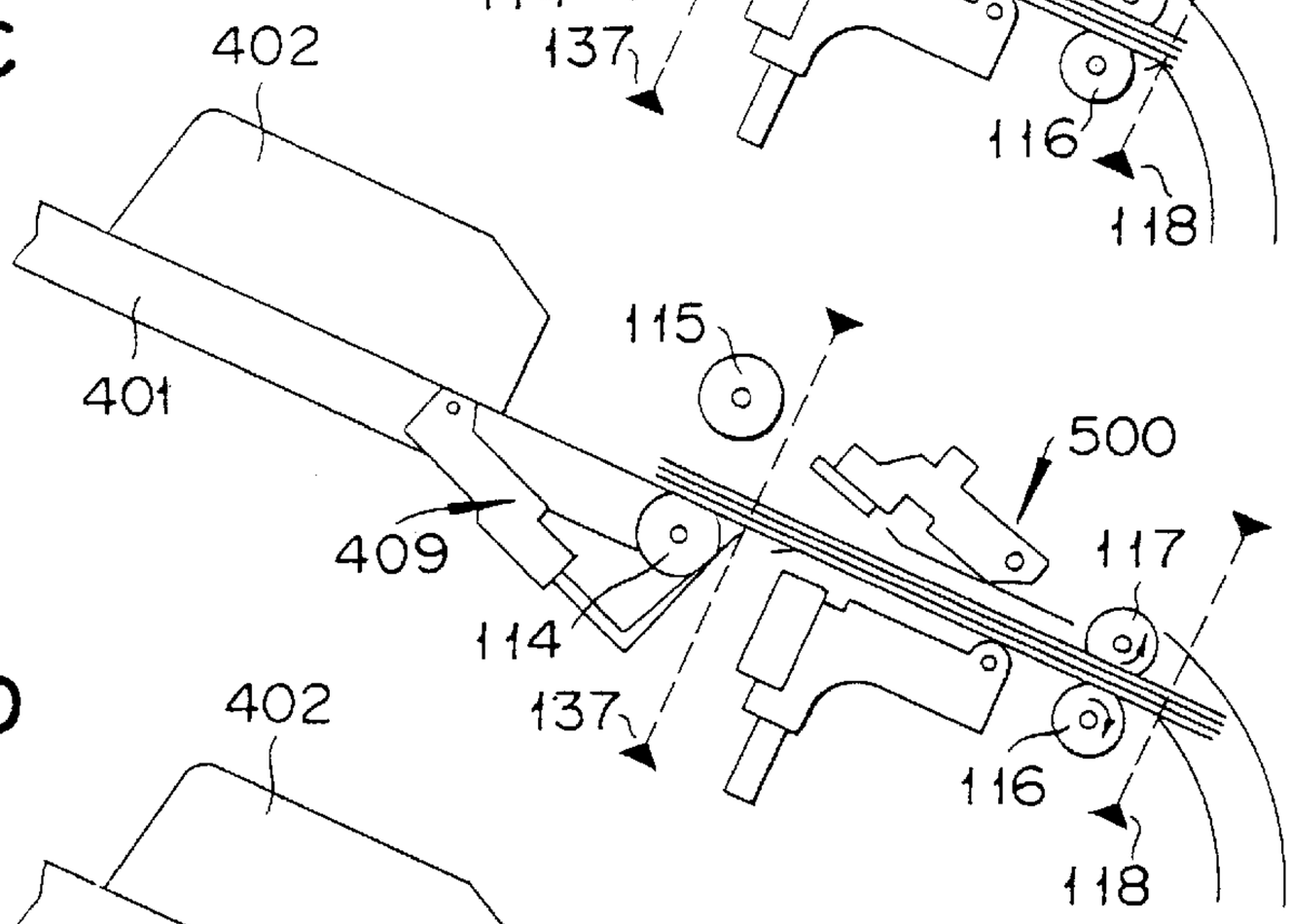


FIG. 30D

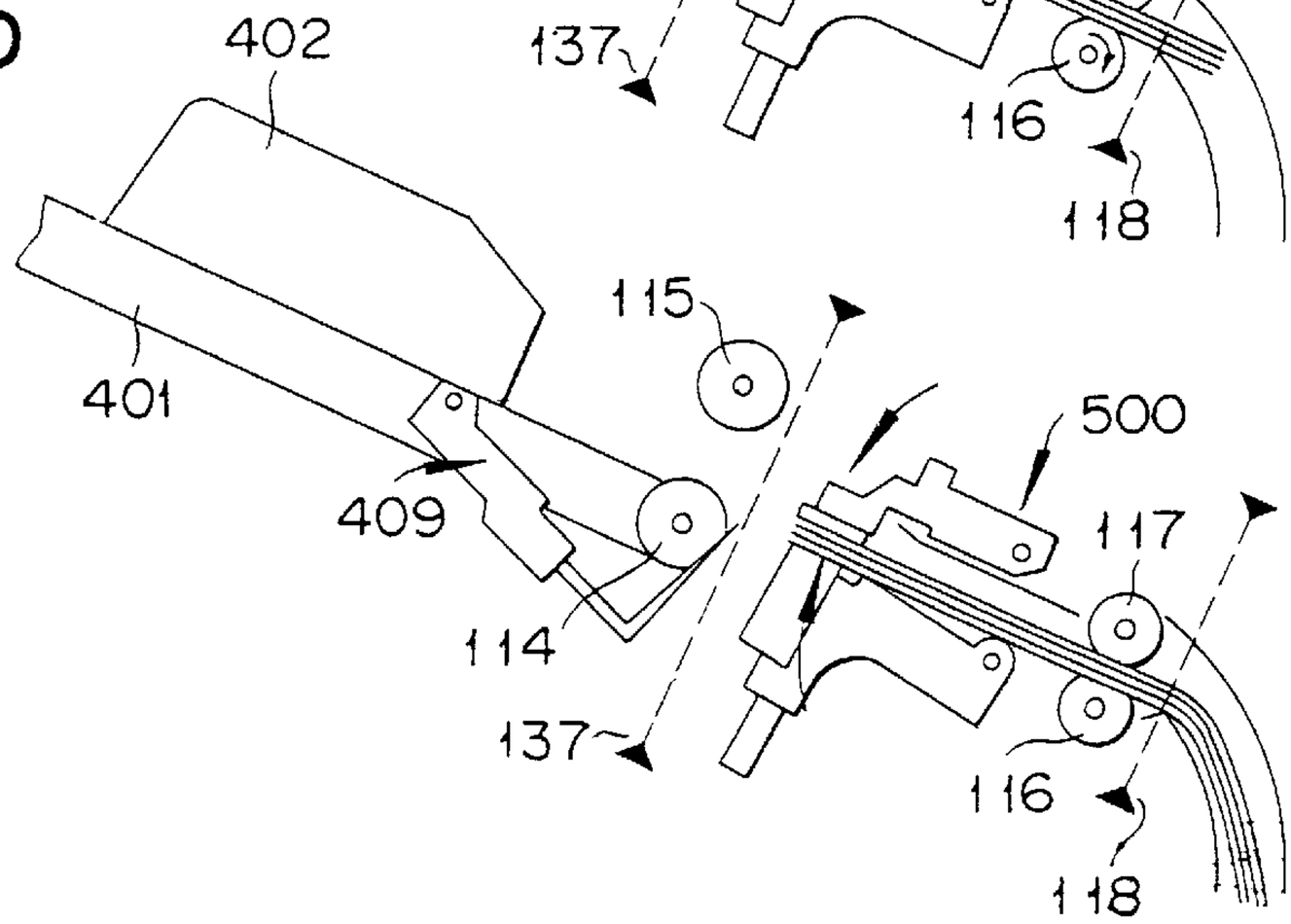


FIG. 31

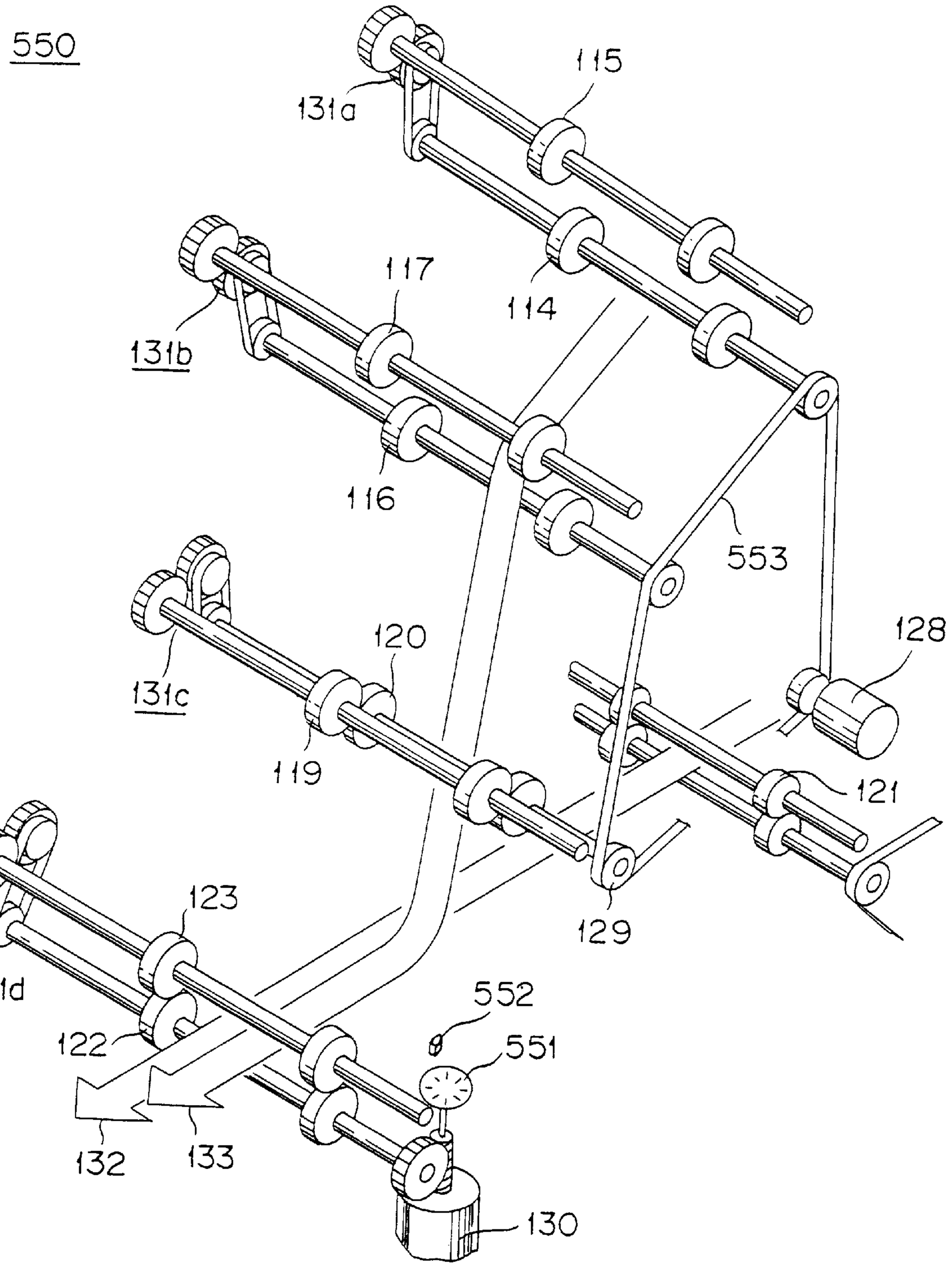

550
131a
115
117
114
131b
116
553
120
128
131c
119
121
123
131d
552
551
122
132
133
130

FIG. 32

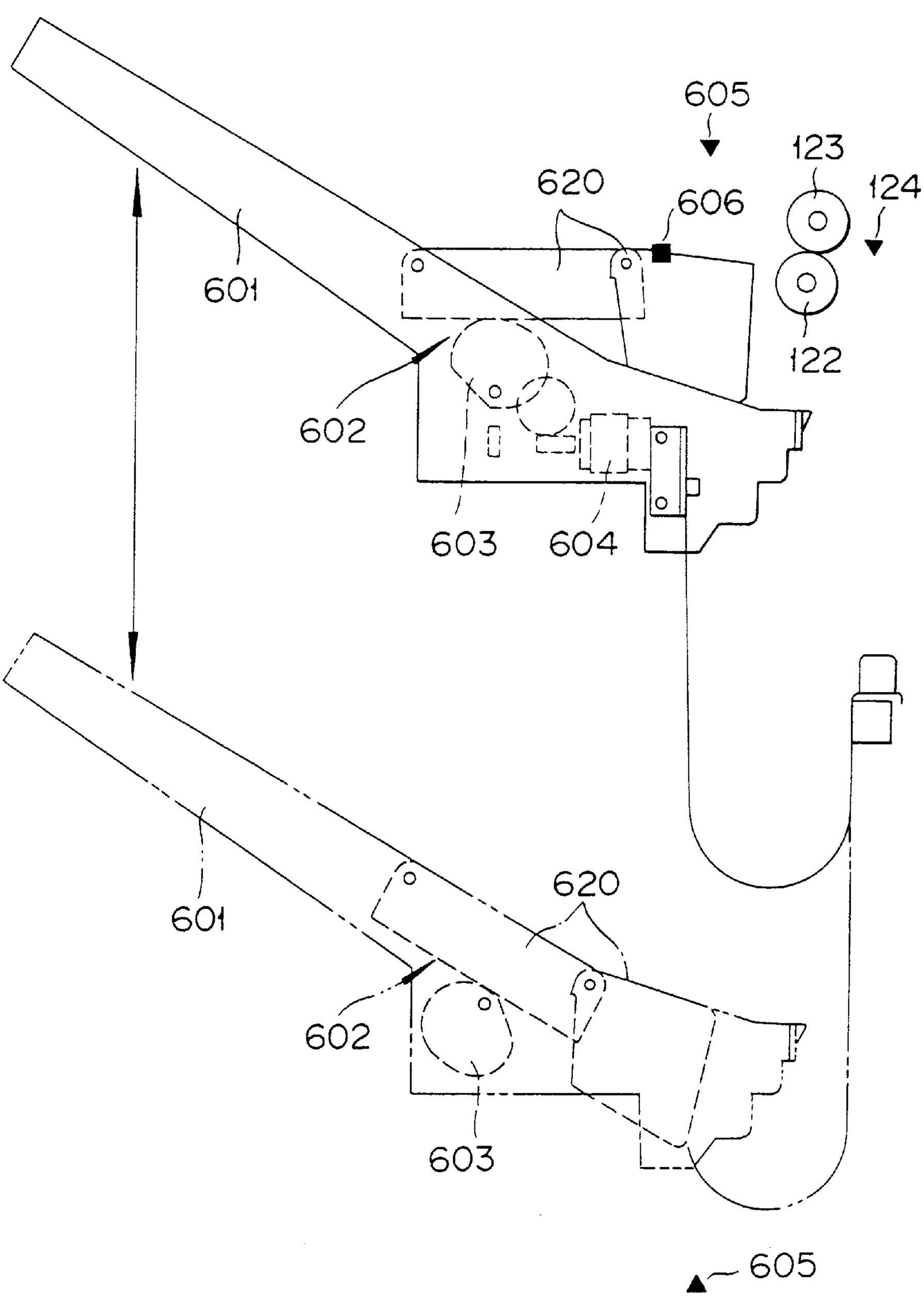


FIG. 33

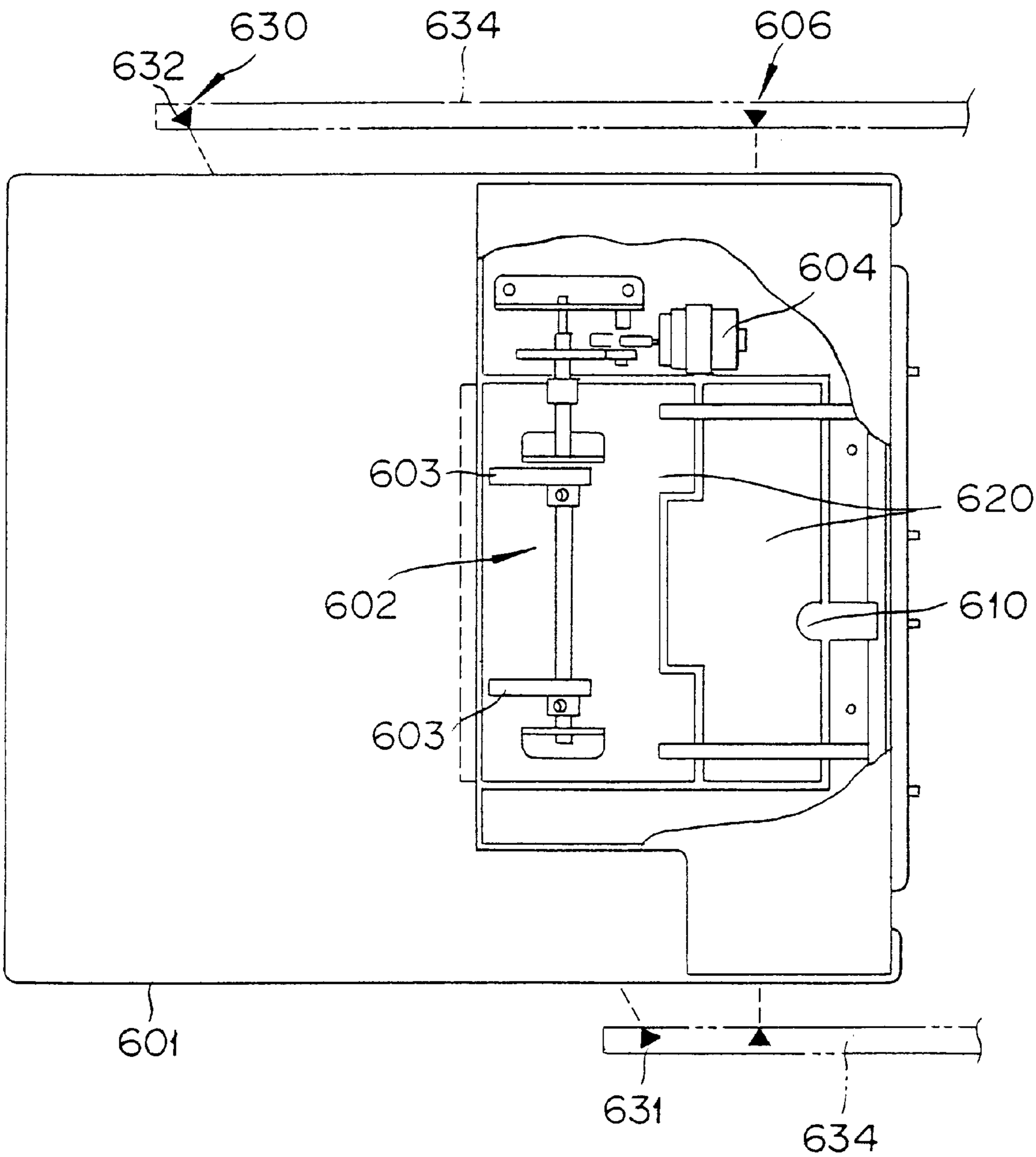


FIG.34A

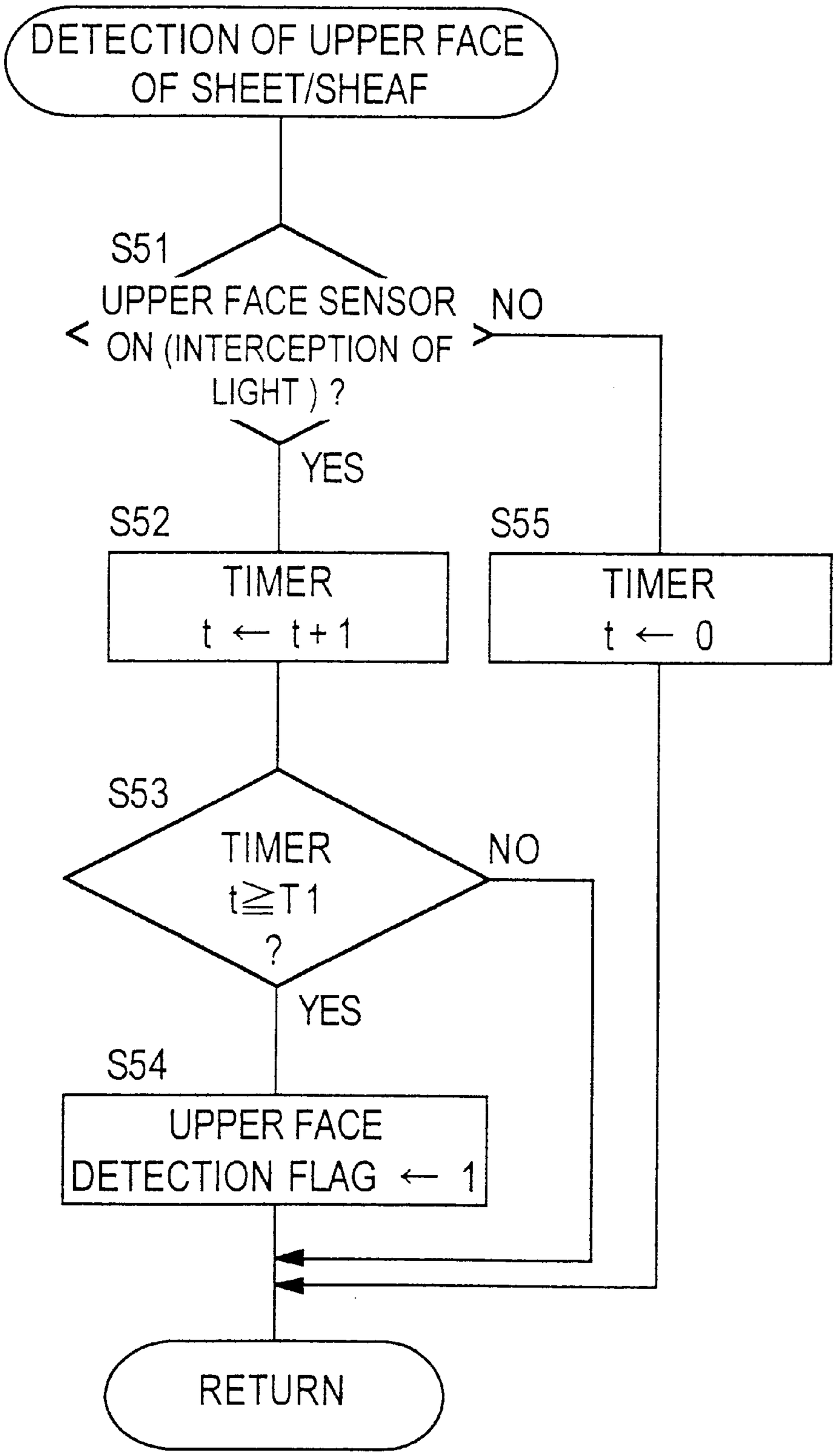

DETECTION OF UPPER FACE OF SHEET/SHEAF
S51
UPPER FACE SENSOR ON (INTERCEPTION OF LIGHT) ?
NO
YES
S52
TIMER
t ← t+1
S55
TIMER
t ← 0
S53
TIMER
t≧T1
?
NO
YES
S54
UPPER FACE DETECTION FLAG ← 1
RETURN

FIG. 34B

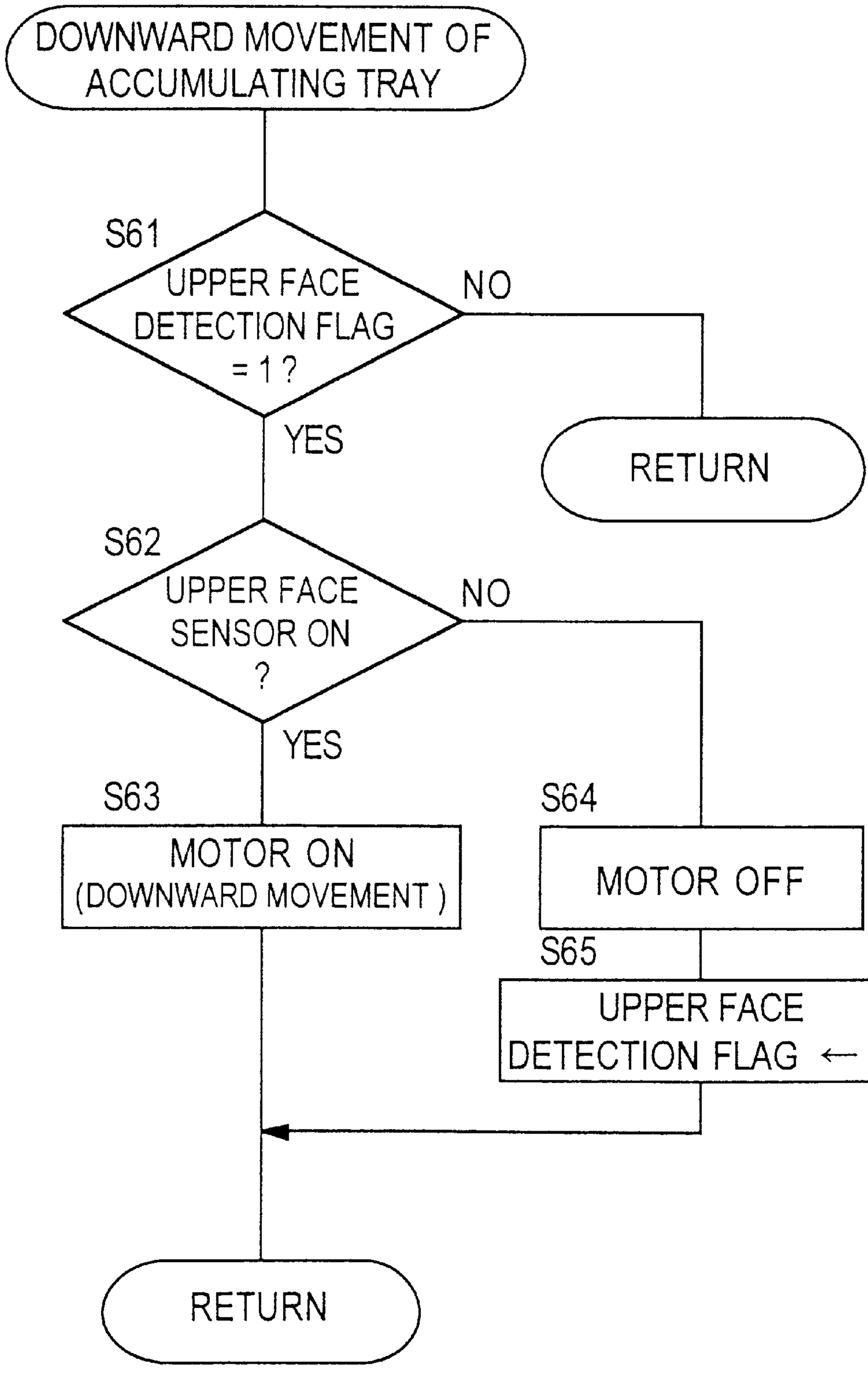

DOWNWARD MOVEMENT OF ACCUMULATING TRAY
S61
UPPER FACE DETECTION FLAG = 1 ?
NO
YES
RETURN
S62
UPPER FACE SENSOR ON ?
NO
YES
S63
MOTOR ON (DOWNWARD MOVEMENT)
S64
MOTOR OFF
S65
UPPER FACE DETECTION FLAG ← 0
RETURN

FIG. 35A

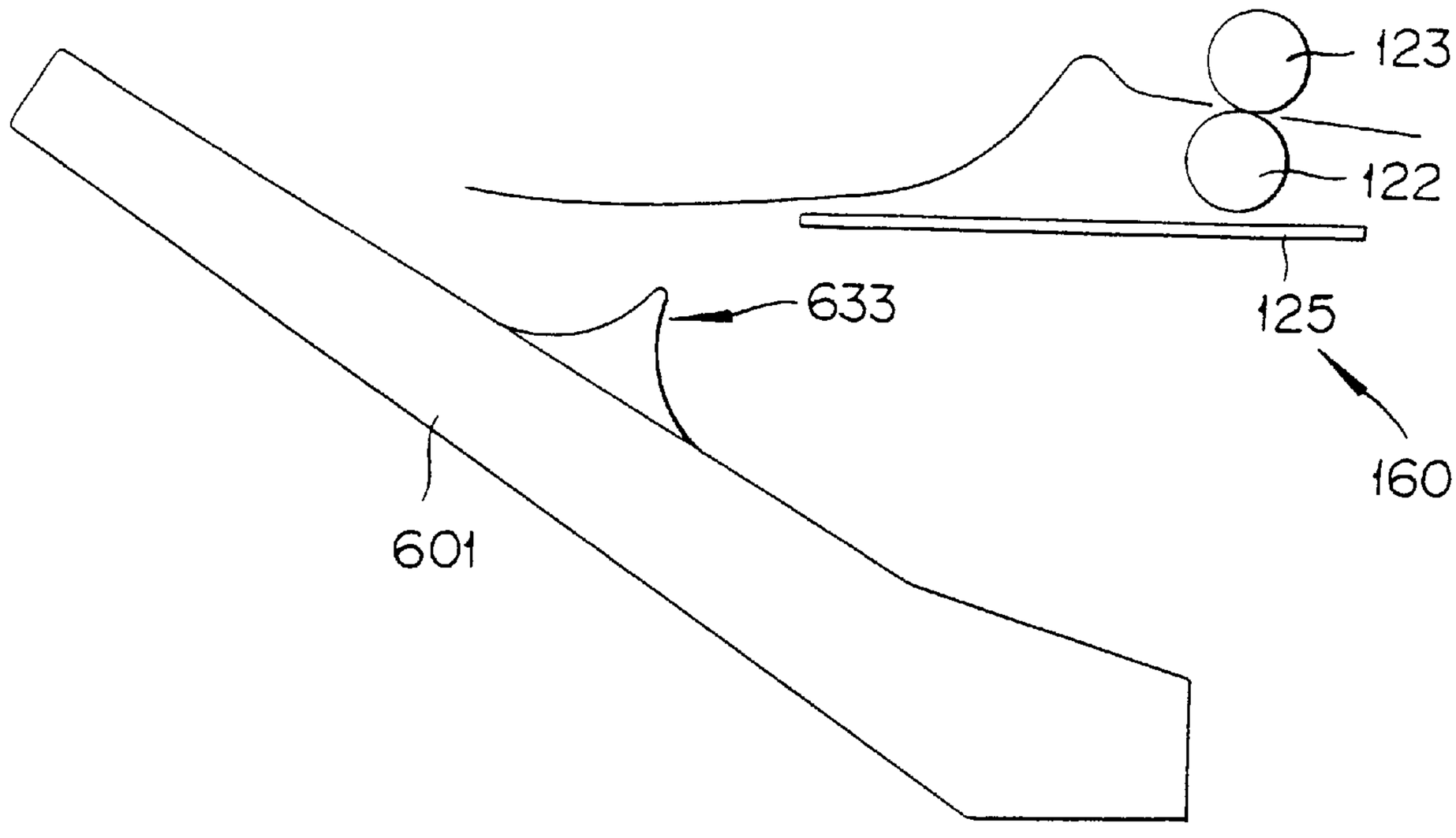


FIG. 35B

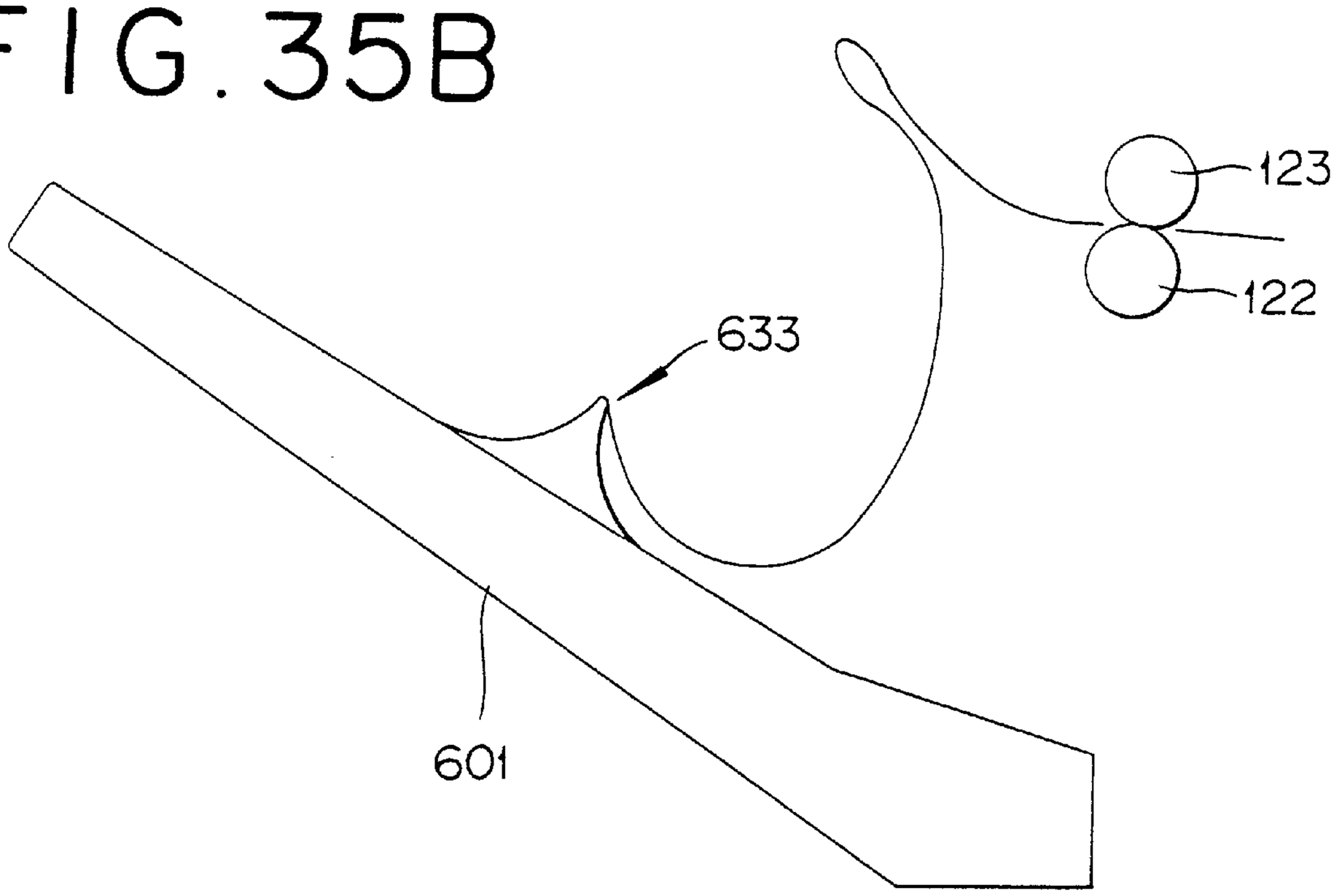


FIG. 36

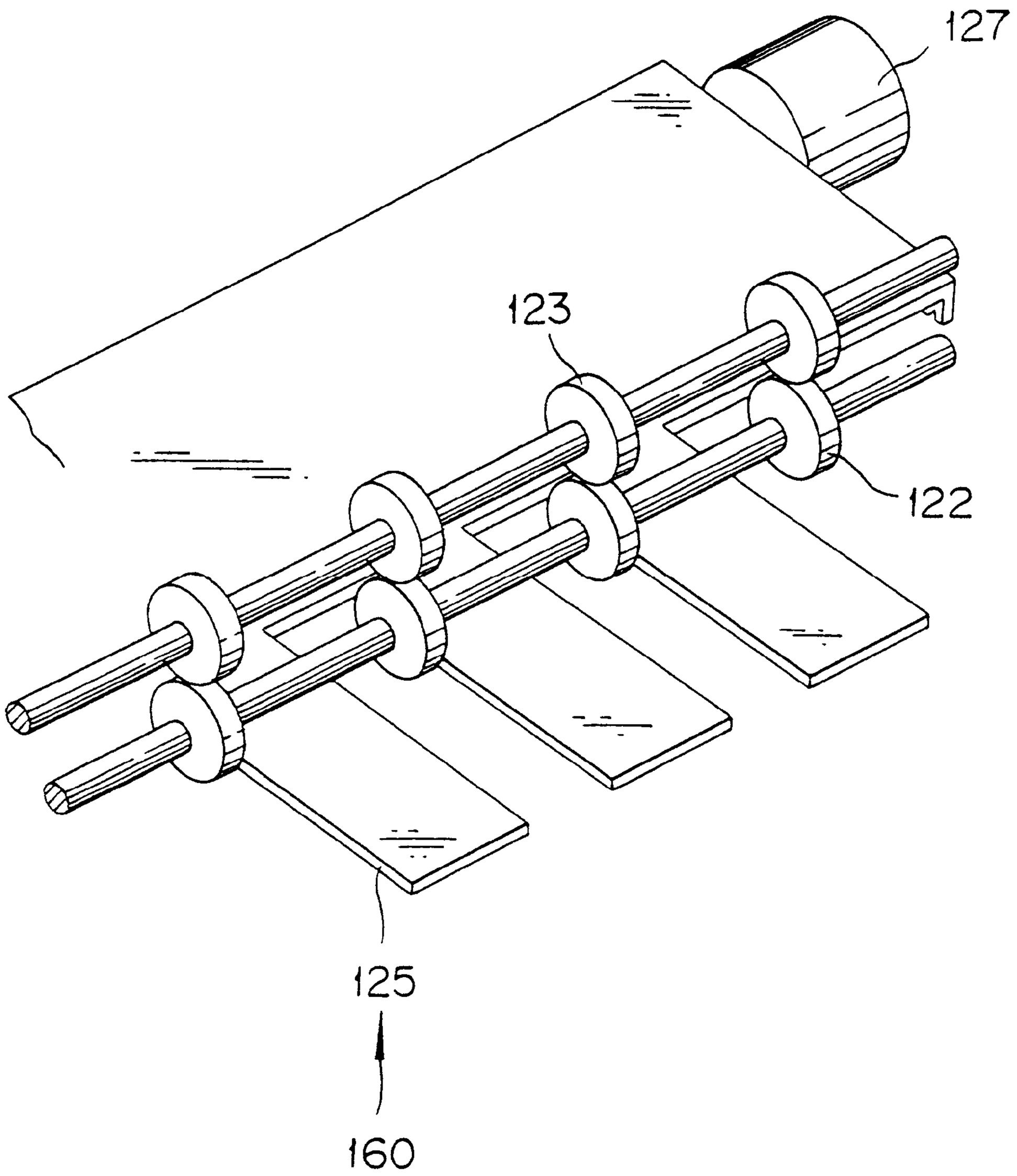


FIG.37

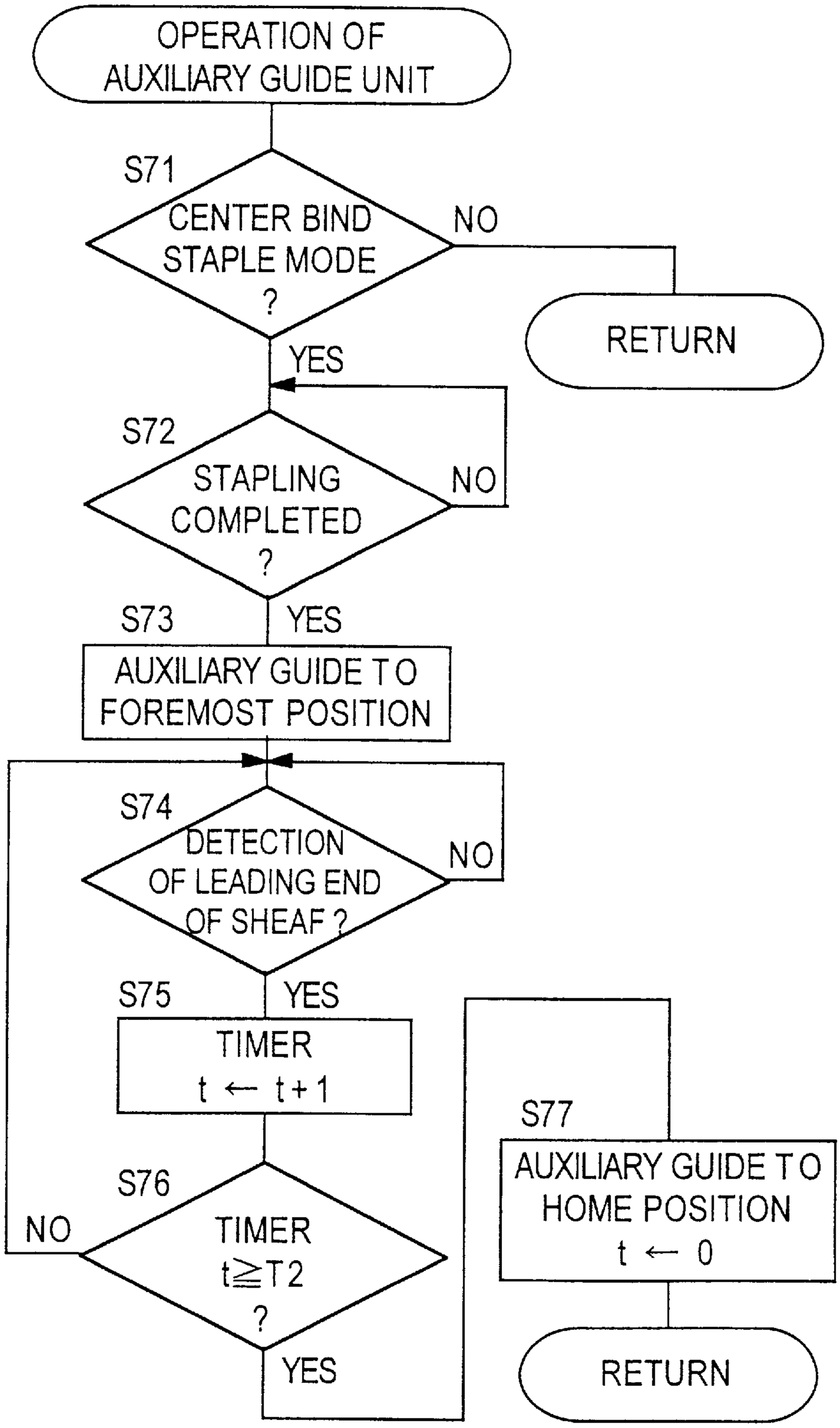

OPERATION OF AUXILIARY GUIDE UNIT
S71
CENTER BIND STAPLE MODE ?
NO
RETURN
YES
S72
STAPLING COMPLETED ?
NO
S73
YES
AUXILIARY GUIDE TO FOREMOST POSITION
S74
DETECTION OF LEADING END OF SHEAF ?
NO
S75
YES
TIMER
t ← t+1
S76
NO
TIMER
t≧T2
?
YES
S77
AUXILIARY GUIDE TO HOME POSITION
t ← 0
RETURN

FIG. 38

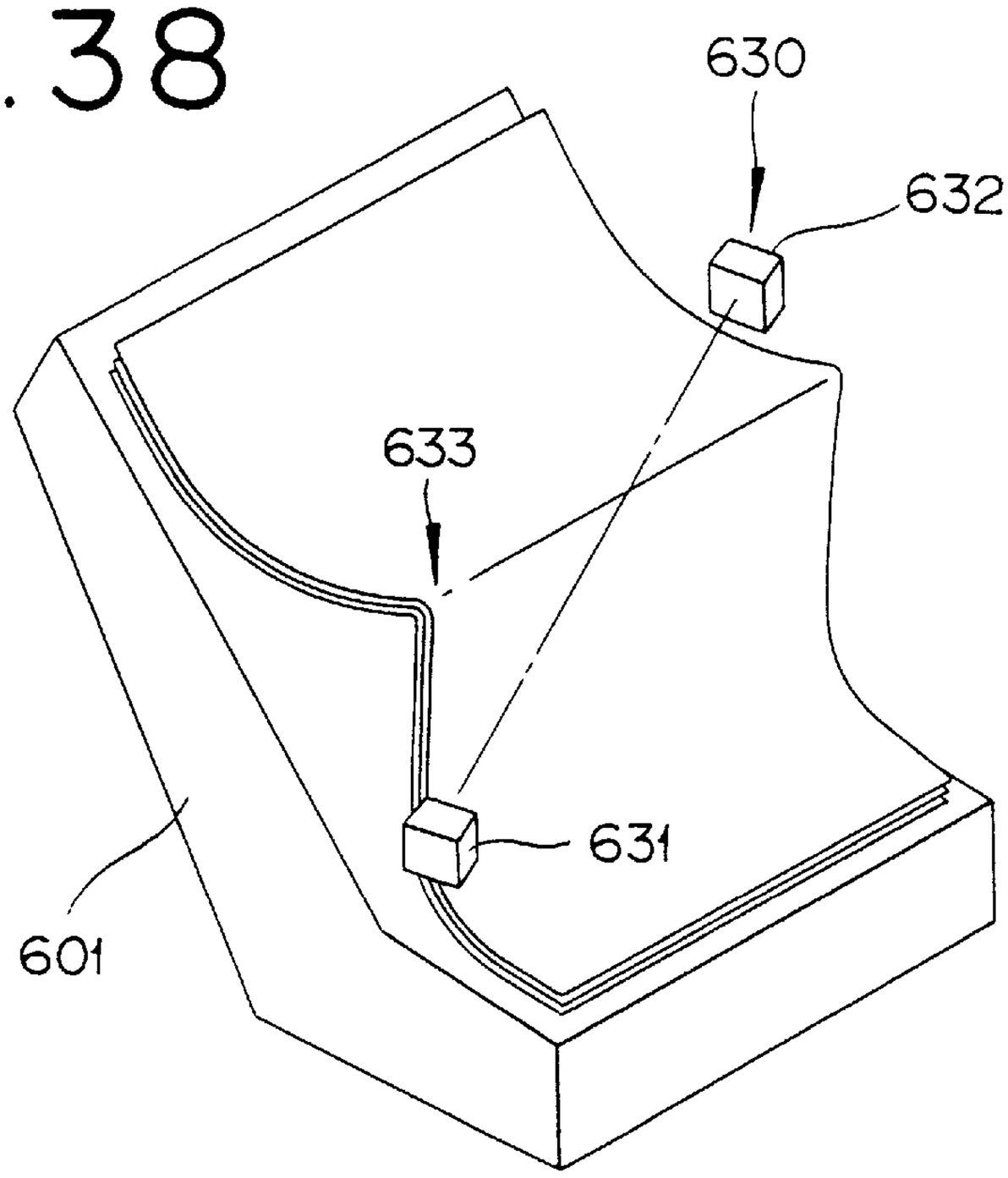


FIG. 39

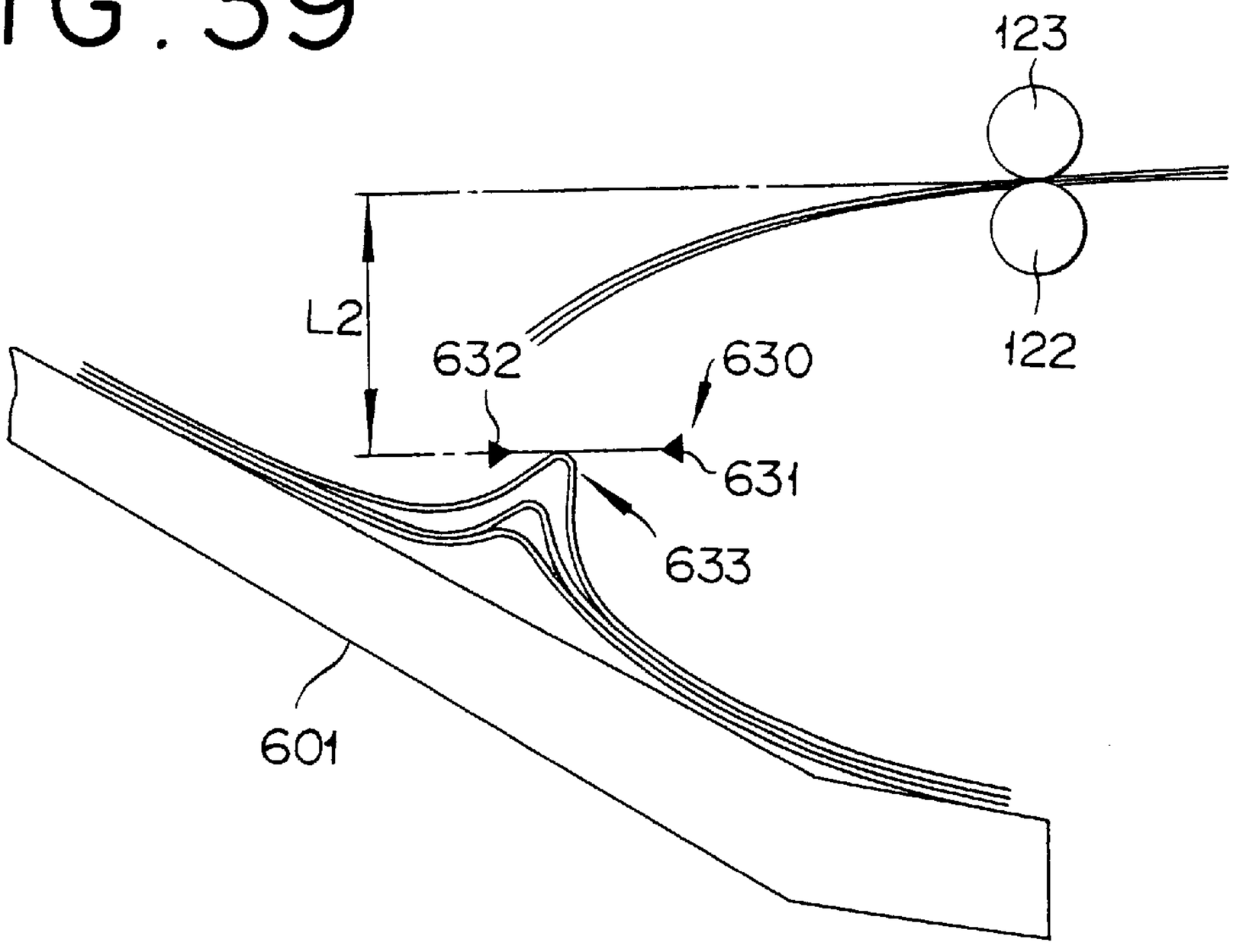
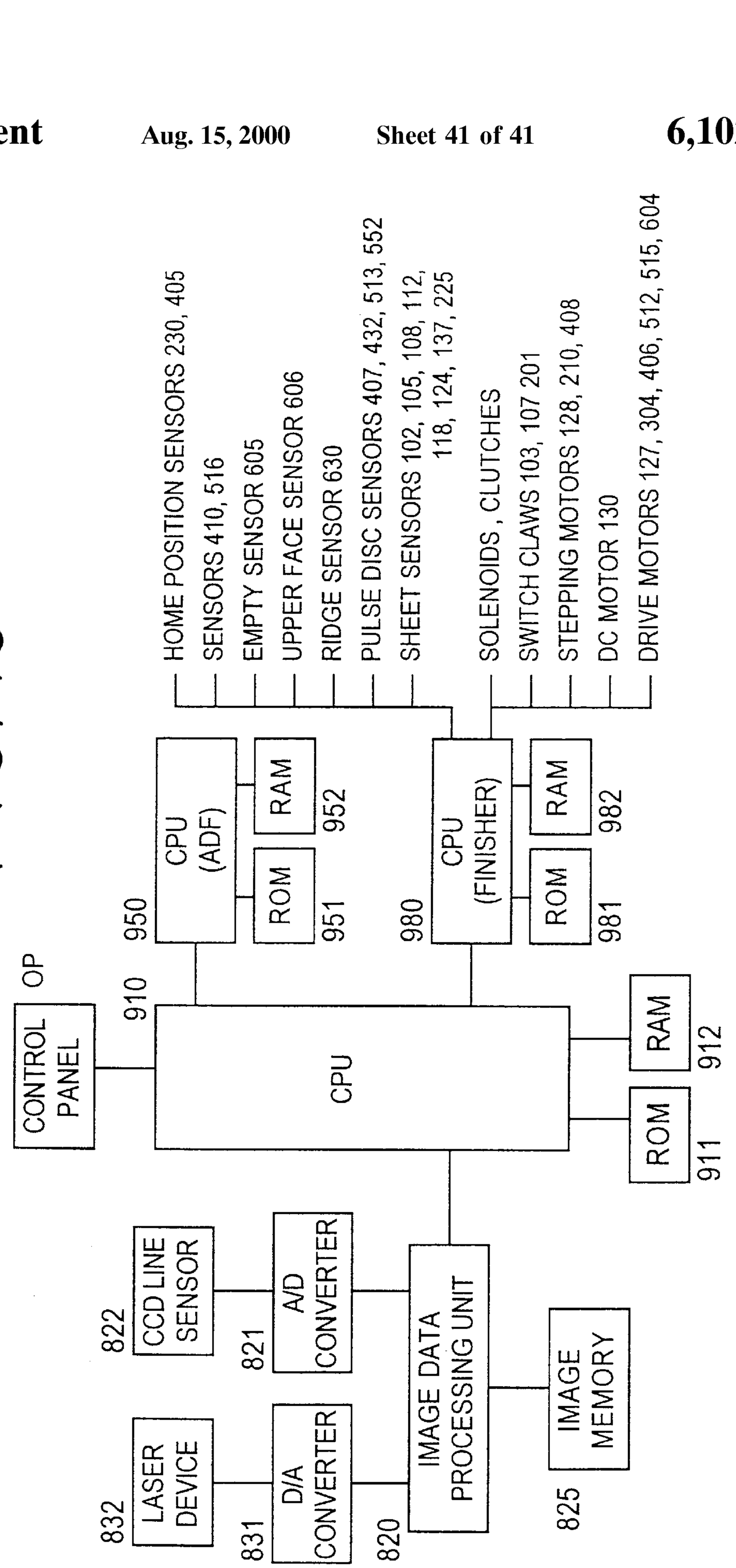


FIG. 40


HOME POSITION SENSORS 230, 405
SENSORS 410, 516
EMPTY SENSOR 605
UPPER FACE SENSOR 606
RIDGE SENSOR 630
PULSE DISC SENSORS 407, 432, 513, 552
SHEET SENSORS 102, 105, 108, 112, 118, 124, 137, 225
SOLENOIDS, CLUTCHES
SWITCH CLAWS 103, 107 201
STEPPING MOTORS 128, 210, 408
DC MOTOR 130
DRIVE MOTORS 127, 304, 406, 512, 515, 604
950
CPU (ADF)
ROM
951
RAM
952
980
CPU (FINISHER)
ROM
981
RAM
982
OP
CONTROL PANEL
910
CPU
RAM
912
ROM
911
822
CCD LINE SENSOR
821
A/D CONVERTER
IMAGE DATA PROCESSING UNIT
IMAGE MEMORY
825
832
LASER DEVICE
831
D/A CONVERTER
820

FINISHER

BACKGROUND OF THE INVENTION

1. Field of the Invention

This invention relates to a finisher which gives such additional-workings as creasing, folding, punching, stapling and binding to sheets outputted from such image forming devices as printers and copying machines. More particularly, this invention relates to a finisher which gives the sheets the additional-workings, which include at least the mode of folding a sheet to a size smaller than the size of the sheet in the unfolded state, and aligns a sheaf of stacked sheets in a receiving tray unit which stores and stacks sheets.

2. Description of the Related Art

Recently, various finishers which give various additional-workings to sheets with an image formed surface which are outputted from such image forming devices as printer and copying machines, have been proposed. The term "additional-workings" as used herein means various working processes such as sorting sheets, filing sheets with staples, folding sheets in two (double-folding), creasing sheets (creasing), or folding sheets in three or in a cross section like a letter Z (Z-folding), binding sheets with mucilage, and punching sheets for filing. The finisher generally is provided with a receiving tray unit for temporarily storing sheets that have been folded and punched. The sheaf of sheets, which has been stacked and aligned in the receiving tray unit, is conveyed to a stapler and is stapled.

In the field of ordinary printers and copying machines, the end faces of the sheets are aligned along the conveying direction. The alignment is attained by utilizing the weight of sheet, having a transmission gear, or adjusting the angle of a tray U.S. Pat. No. 4,905,053, for example).

The conventional finisher with a stapling mechanism and a sheet folding mechanism aligns a sheet in the conveying direction without reference to the presence or absence of a sheet folding.

The alignment utilizing the weight of a sheet, therefore, encounters many unstable factors and tends to produce an imperfect result.

The alignment which uses a transmission gear only necessary to a small sheet or a folded sheet has such problems as boosting production cost and electric power consumption, and suffering degradation of productivity because of the inability to quickly do the series of additional-workings due to the addition of the transmission gear.

Further, the folded sheet has an unstable shape as compared with an equivalent sheet in an unfolded state because of the presence of the crease of fold. The alignment which uses a tray capable of varying the working angle and necessary only for aligning a folded sheet with an unstable shape, has the problem of boosting manufacturing cost and electric power consumption due to the additional device for the adjustment of the tray angle.

SUMMARY OF THE INVENTION

An object of this invention is to provide a finisher which is capable of steadily aligning a sheaf even when the sheaf include folded sheets. Another object of this invention is to provide a finisher which reduced production cost and electric power consumption, and refrains from impeding productivity.

To accomplish the objects, this invention concerns a finisher for stacking sheets folded to a size smaller than the size of an equivalent sheet in an unfolded state, the finisher

comprising: a receiving tray unit stacking sheets, a first regulating device disposed as projected from the receiving tray unit and contacting to one end face of a sheaf stacked in the receiving tray unit, and a second regulating device disposed as projected from the receiving tray unit and contacting to leading ends of sheets fed into the receiving tray unit, which presses the other end face of the sheaf and aligns one end face with the first regulating device, the second regulating device being actuated, when the sheaf include sheets folded by a fold mode for folding sheets to a size smaller than the size of an equivalent sheet in an unfolded state, to move and push the folded sheets into the first regulating device.

In the finisher, when one sheet is conveyed in the direction of the receiving tray unit, the leading end of this sheet contacts to the second regulating device and springs back toward the first regulating device and quickly falls onto the receiving tray unit or onto the uppermost one of the sheets already held therein. As a result, the sheets can be aligned optimally and quickly, and the temporary accumulation of sheets in the receiving tray unit can be completed early even when the sheets are fed and outputted with a short interval. Owing to this merit, the finisher is enabled to carry out the series of additional-workings expeditiously and enjoy improved productivity. Moreover, the finisher utilizes the spring-back force which the sheet produces on contacting to the second regulating device. It reduces misalignment of the sheet as compared with the alignment which resorts solely to the weight of sheet. When the sheaf include folded sheets, the second regulating device is actuated to move and push the folded sheets into the first regulating device. In other words, the receiving tray unit effects stable alignment by considering the fact that the sheaf includes folded sheets. Moreover, the alignment does not utilize a transmission gear, and the productivity is not decreased. The addition of a transmission gear or a device for adjusting the tray angle is not required. It prevents manufacturing cost or electric power consumption from increasing.

The workings performed in the finisher include a double-folding and a Z-folding.

The finisher is connected to an image forming device for forming a image on a sheet and performs various additional-workings on a sheet having an image formed surface which has been outputted from the image forming device. The work of sheet folding is performed in the finisher and the additional-workings include a stapling and a binding.

The second regulating device is preferably disposed on a side of the receiving tray unit where a crease of the sheets folded by the fold mode is located, and is formed in a shape curved toward the first regulating device. To be specific, the second regulating device has a radius or is rounded, or has a bent shape.

When folded sheets are stacked, the resultant sheaf of the sheets is not parallel to the stacking face of the receiving tray unit but inclines toward the first regulating device side from the second regulating device side. The inclination sharply increases in response to the number of folded sheets stored in the receiving tray unit. If the second regulating device is so shaped as to perpendicularly intersect the stacking face of the receiving tray unit, it cannot satisfactorily align sheets in the upper part of the sheaf. In contrast, the second regulating device according to the invention is formed in a curved shape. Consequently, the alignment of the sheaf including folded sheets is done uniformly and perfectly throughout from the lower part to the upper part of the sheaf.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. **1** is a schematic explanatory cross section illustrating an embodiment having a finisher according to this

invention connected to a copying machine as an image forming device;

FIG. **2** is a schematic structural diagram illustrating the essential section of the finisher;

FIG. **3** is a cross section illustrating the construction of a folding device;

FIG. **4** is a cross section illustrating the folding device which is jammed;

FIG. **5**A and FIG. **5**B are cross sections illustrating the essential section of a mechanism for regulating the first folding position in the folding device;

FIG. **6** is a bottom view illustrating the mechanism for regulating the first folding position in the folding device;

FIG. **7** is a perspective view illustrating the essential section of a first folding stopper;

FIG. **8** is a cross section illustrating the state of the folding device under the A3 Z-folding mode;

FIG. **9** is a cross section illustrating the state of the folding device under the A3 double-folding mode;

FIG. **10** is a cross section illustrating the state of the folding device under the creasing mode;

FIG. **11** is a flow chart illustrating a process for setting a sheet conveying path;

FIG. **12** is a flow chart illustrating a process for retracting the first folding stopper during the restoration from a sheet jam;

FIG. **13** is a perspective view illustrating a punching device;

FIG. **14** is a side view illustrating the punching device;

FIG. **15** is a cross section illustrating the construction of an additional-work tray unit;

FIG. **16** is a lateral cross section illustrating an additional-working tray of the additional-work tray unit;

FIG. **17** is a partially cutaway bottom view illustrating the additional-working tray of the additional-work tray unit;

FIG. **18**A–FIG. **18**C are explanatory diagrams illustrating steps for aligning sheets in the additional-work tray unit and FIG. **18**D is an explanatory diagram illustrating steps for conveying a sheaf of stacked and aligned sheets in the direction of a stapler;

FIG. **19**A–FIG. **19**C are diagrams illustrating various stapling modes;

FIG. **20** is a flow chart illustrating the control of motion of a trailing end stopper;

FIG. **21** is a flow chart illustrating the operation of a first sheet-conveying roller during the sheet alignment;

FIGS. **22**A and **22**B are explanatory diagrams illustrating the operation of aligning a sheaf including Z-folding sheets;

FIG. **23** is a structural diagram illustrating a stapler together with a second sheet-conveying roller as well as the first sheet-conveying roller;

FIG. **24** is a schematic perspective view illustrating the construction of the stapler;

FIG. **25**A–FIG. **25**C are structural diagrams illustrating the first sheet-conveying roller;

FIG. **26** is an explanatory diagram illustrating a portion defined as a sheet position deviation;

FIG. **27**A is a graph showing the relation between the presence or absence of "forced-parallel movement" and the sheet position deviation, and FIG. **27**B is a graph showing the relation between hardness of the sheet-conveying rollers and the sheet position deviation;

FIG. **28**A–FIG. **28**F are explanatory diagrams illustrating the operation of leading end binding;

FIG. **29**A–FIG. **29**D are explanatory diagrams illustrating the operation of intermediate binding;

FIG. **30**A–FIG. **30**D are explanatory diagrams illustrating the operation of trailing end binding;

FIG. **31** is a perspective view illustrating an artist concept of a sheet discharge unit for conveying a stapled sheaf and one unstapled sheet in the direction of an accumulating tray unit;

FIG. **32** is a structural diagram illustrating the accumulating tray unit;

FIG. **33** is a partially cutaway bottom view illustrating an accumulating tray of the accumulating tray unit;

FIG. **34**A is a flow chart illustrating a control routine for the detection of the upper face of sheets (sheaf) in a series of operations of the accumulating tray unit, and FIG. **34**B is a flow chart illustrating the control routine for moving the accumulating tray downward with a drive motor in the series of operations of the accumulating tray unit;

FIG. **35**A is a schematic structural diagram illustrating an auxiliary guide of a guide unit, and FIG. **35**B is an explanatory diagram illustrating failed discharge of a sheaf like a weekly magazine in which the sheets are folded in two and the creases are bound;

FIG. **36** is a perspective view illustrating the auxiliary guide;

FIG. **37** is a flow chart illustrating steps for the operation of the guide unit;

FIG. **38** is a schematic perspective view illustrating a ridge sensor provided in the accumulating tray unit;

FIG. **39** is a diagram illustrating the state on which a weekly-magazine-like sheaf is stored; and

FIG. **40** is a block diagram illustrating a control system for controlling the various works or operation.

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

The preferred embodiments of this invention will be described below with reference to the accompanying drawings.

FIG. **1** is a schematic explanatory cross section illustrating an embodiment having a finisher **100** according to this invention connected to a copying machine **10** as an image forming device and FIG. **2** is a schematic structural diagram illustrating the essential section of the finisher **100**.

In this specification, the direction of conveyance of a sheet will be referred to as "conveying direction" and the direction perpendicular to the conveying direction as "orthogonal direction." Then, the orientations of a sheet are defined as follows relative to the conveying direction. The orientation of the sheet whose longitudinal direction falls along the conveying direction will be referred to as "longitudinal" and the orientation of the sheet whose longitudinal direction perpendicularly crosses the conveying direction as "lateral."

COPYING MACHINE **10**

The illustrated copying machine **10** to which the finisher **100** is connected is what is called a digital copying machine. The digital copying machine reads and temporarily stores in a memory an image on the surface of a document and, when necessary, executes various image processings. Then, it forms the image on a sheet by the well-known electropho-

tographic method and outputs sheets with the copied image one by one from a sheet output section **10***b*.

The copying machine **10** has an automatic document feeder **12** (hereinafter referred to as "ADF")on the upper section. The ADF **12** feeds one document or a plurality of documents (group of documents) set on a tray **14** one by one onto a platen glass (not shown) of the copying machine **10** and, after scanning the image, outputs and stacks the document onto a tray **16**.

The copying machine **10** of the present embodiment is a so-called first page system which starts a copying motion from the first page onward of the group of documents. On the tray **14** of the ADF **12**, the group of documents are set, with the first page turned upward. The copying machine of the first page system obviates the necessity for inputting or detecting the number, odd or even, of the documents in the group as when an image on one side of the document is copied on the obverse and reverse sides of one sheet. It produces advantages such as a quick copying motion.

As the document is set on the platen glass as by the ADF **12**, the image on the document is read by an image reader (not shown) built in the copying machine **10**, converted into digital data, and stored in a memory of the control unit. The copying operation, after read out of the image data, is executed as combined with such necessary editorial processings as, for example, changing the order of pages, inverting an image, or producing copied images on both sides of a sheet.

This copying machine **10** is provided near the sheet output section **10***b* with a turn-back mechanism **20** for turning a sheet with copied image upside down. This mechanism will be described more specifically herein below.

<<General construction and general operation of finisher **100**>>

[General construction]

The finisher **100** of the present embodiment performs, either selectively or as suitably combined, such a folding work as folding the sheets outputted from the sheet output section **10***b* of the copying machine **10** and conveyed one by one, in two or three (Z-folding in a cross section like a letter Z) as occasion demands, a punching work for forming holes for filing in the edges of the sheets, and a stapling for binding a sheaf with staples. Further, in this finisher **100** the mode of conveyance of sheets, the mode of stacking of sheets, or the mode of folding of sheets are designed on the assumption that it will be used as connected to the copying machine or a printer as an image forming device of the first page system.

The finisher **100**, as illustrated in FIG. **2**, comprises a feed channel section **150** through which a sheet P outputted from the sheet output section **10***b* is fed, a folding device **200** which folds or creases the sheets conveyed one by one, a punching device **300** which forms holes for filing in the sheets P conveyed one by one, an additional-work tray unit **400** which stacks and aligns the sheets before a stapling work, a stapler **500** disposed on the downstream side of the additional-work tray unit **400** and stapling a sheaf of stacked and aligned sheets, an accumulating tray unit **600** which is capable of receiving a stapled sheaf or an unstapled sheet, and an output tray unit **110** which receives the sheets outputted from the finisher **100**.

The feed channel section **150** is provided with a conveying roller **101** and a guide plate. The folding device **200** is provided with a plurality of folding rollers **207**, **208**, and **209** and is adapted to nip a sheet P between the folding rollers **207**, **208**, and **209** and folds or creases the sheet P. The stapler **500** is so constructed as to be moved in the two directions, i.e. the conveying direction and the orthogonal direction of the sheaf stacked and aligned in the additional-work tray unit **400**.

For the purpose of conveying the sheet to various sections in the finisher **100**, conveying rollers **104**, **106**, **111**, and **121** are disposed along the sheet conveying paths. For the purpose of conveying the sheaf, sheaf-conveying rollers **114** and **115**, **116** and **117**, and **119** and **120** are disposed along the conveying paths of the sheaf. A discharge roller **109** for discharging the sheet P into the output tray unit **110**, a discharge roller **113** for discharging the sheet P into the additional-work tray unit **400**, and discharge rollers **122** and **123** for discharging the sheet P or the sheaf into the accumulating tray unit **600** are respectively disposed at the terminal positions of the conveying paths.

For the purpose of changing the destination of the sheet being conveyed, a plurality of switch claws **201**, **103** and **107** are disposed on the sheet conveying paths. The switch claw **201**, which is disposed between the feed channel section **150** and the folding device **200**, decides whether or not the sheet P is fed into the folding device **200**. The punching device **300** is disposed on the downstream side of the switch claw **201** and is enabled to punch the sheet conveyed from the feed channel section **150** or the sheet conveyed from the folding device **200**. The switch claw **103** disposed on the downstream side of the punching device **300** directs whether the sheet P is conveyed to the output tray unit **110** or to the additional-work tray unit **400** or the sheet P is directly conveyed to the accumulating tray unit **600**. The switch claw **107** disposed on the downstream side of the switch claw **103** directs whether the sheet P is conveyed to the output tray unit **110** or to the additional-work tray unit **400**.

For the purpose of timing the driving or stopping the various components in the finisher **100**, a plurality of sensors **102**, **105**, **108**, **112**, **118**, **124** and **225** for detecting the sheet are disposed on the sheet and sheaf conveying paths.

The finisher **100** of the present embodiment is further provided with a guide unit **160** for preventing the sheaf bound by stapling like a weekly magazine from being defectively discharged into the accumulating tray unit **600**. The guide unit **160** illustrated in the diagram is composed of an auxiliary guide **125** which supports the lower side of the sheaf discharged from a space between discharge rollers **122** and **123**, and is allowed freely to advance and retract. This construction permits the leading end of the sheaf being discharged to fall toward the downstream side along the discharging direction, further than the peak of the formerly discharged center bound sheaf even when the sheaves of sheets are stacked such that the bound sections project upward like a mountain. This results in precluding the possibility of the leading ends of the successively discharged sheaves being caught in the neighborhood of the peaks of the already stacked sheaves.

[General operation]

The finisher **100** is capable of performing a plurality of additional-workings (folding, punching and stapling) on the sheets. The user of the finisher **100** may select freely these works by the use of a control panel of the copying machine **10**.

When the user selects a mode excluding a stapling, the sheet P discharged from the sheet output section **10***b* of the copying machine **10** is worked by the folding device **200** and the punching device **300** in response to instructions of the user and conveyed by means of rollers to the output tray unit **110** or the accumulating tray unit **600** for storage.

When the user selects a mode including a stapling, first the sheet P is worked by the folding device **200** and the

punching device **300** in response to instructions of the user as similarly to the mode excluding the stapling. Then, a certain number of sheets P which have been folded and/or punched are conveyed to the additional-work tray unit **400** and sequentially stacked and aligned Thereafter, the sheets which have been stacked and aligned are fed as one sheaf by rollers to the stapler **500**.

After the stapler **500** has bound the sheaf by driving staples in the sheaf at the positions selected by the user, the stapled sheaf is conveyed by the rollers to the accumulating tray unit **600** and is stored.

In this finisher **100**, the folding device **200** and the punching device **300** (as means working the incoming sheets one by one) are disposed on the upstream sides of the position of the switch claw **103**, or on the upstream sides of the branching points of the conveying paths to a plurality of receiving tray units (referring collectively to the output tray unit **110**, the additional-work tray unit **400**, and the accumulating tray unit **600**). The sheets which have undergone the works (folding and punching in this embodiment) one by one, therefore, can be discharged to any of the receiving tray units.

The main mechanisms of the finisher **100** will be sequentially described in detail below.

<<Folding device **200**>>

FIG. **3** is a cross section illustrating the construction of the folding device **200**, FIG. **4** is a cross section illustrating the folding device **200** which is jammed, FIGS. **5**A and **5**B and FIG. **6** are respectively cross sections and a bottom view illustrating the essential section of a mechanism for regulating a first folding position in the folding device **200**, and FIG. **7** is a perspective view illustrating the essential section of a first folding stopper.

The folding device **200** is built in the finisher **100** so as to be drawn out toward the front side of the finisher **100** (the foreground side of the face of the sheet bearing FIG. **1**) and is supported as mounted to a rail (not shown) extended in the longitudinal direction of the finisher **100**.

The folding device **200**, as illustrated in FIG. **3**, is composed of a feed channel section **251** for inside feeding a sheet for folding, an adjusting section **252** for correcting the sheet fed into the folding device **200** by removing a deviation, a first conveying section **253** for regulating the first folding position of the sheet conveyed from the adjusting section **252**, a folding section **254** for creasing or folding the sheet, a second conveying section **255** for regulating the second folding position, and a discharging section **256** for conveying the folded sheet from the folding device **200** to the punching device **300**.

[Feed channel section **251**]

The feed channel section **251** comprises the switch claw **201** which selectively guides the sheet to the folding device **200**, conveying rollers **202**, **203** which convey the sheet fed into the folding device **200**, a solenoid (not shown) which rotates the switch claw **201**, and a sheet sensor **225** which detects the sheet fed into the folding device **200**.

[Adjusting section **252**]

The adjusting-section **252** comprises resist rollers **205**, **206** disposed on the downstream side of the feed channel section **251**, a drive motor (not shown) which drives the resist rollers **205**, **206** for folding a sheet, and a solenoid clutch (not shown) which selectively cuts the connection of the motor to the resist rollers **205**, **206**. The resist rollers **205**, **206** are a pair of rollers composed of straight rollers. The surface friction coefficient μ of the roller **205** is set at a level lower than that of the other roller **206**. A guide **260** which is disposed on the upstream side of the resist rollers **205**, **206** is shaped such that the leading end of the sheet is made to contact infallibly to the roller **205** having a lower surface friction coefficient.

The procedure for correcting a deviated sheet is as follows.

First, the sheet sensor **225** detects the leading end of an incoming sheet. At this time, the solenoid clutch is in the OFF state and the driving force of the motor for sheet folding is not transmitted to the resist rollers **205**, **206**.

Then, after the elapse of the time (t+t1) [second], the solenoid clutch is turned on to transmit the driving force to the resist rollers **205**, **206** to convey the sheet to the downstream side. Here, the letter "t" refers to the time [second] required for the leading end of a given sheet to reach the nip part of the resist rollers **205**, **206**.

In consequence of the operation, a loop, V×t1 [mm] (in which V stands for the sheet conveying speed [mm/second]) in length, is formed on the sheet between the conveying rollers **202**, **203** and the resist rollers **205**, **206**. Owing to the formation of this loop, the leading end of the sheet is caused by the intensity of the nerve of the sheet to conform to the contour of the nip part and the deviation of the sheet is adjusted.

[First conveying section **253**]

The first conveying section **253** disposed on the downstream side of the adjusting section **252** comprises first folding stoppers **215**, **216**, **217** and **223** which move into and out of the sheet conveying paths in accordance with the sheet size and the folding, form and regulate the first folding position of the sheet by contacting to the leading end of the sheet, cams **211**, **212** and **213** which actuate the first folding stoppers **215**, **216** and **217**, a stepping motor **210** which rotates the cams **211**, **212** and **213**, and anti-deviation devices **226** of an elastic material which are disposed where the first folding stoppers **215**, **216**, **217** and **223** are contacted to the leading end of the sheet.

The first folding stoppers **215**, **216**, **217** and **223** will be described more specifically herein below. The first folding stopper **217** especially has the function of regulating the first folding position for sheets of two kinds with one stopper.

The three cams **211**, **212** and **213** are fixed to a cam shaft **224** as shifted in angle such that the three first folding stoppers **215**, **216** and **217** are severally moved in and out of the sheet conveying path just once each time the cam shaft **224** produces one complete rotation.

[Folding section **254**]

The folding section **254** disposed between the downstream positions of the resist rollers **205**, **206** and the upstream position of the first folding stopper **215** is possessed of the three folding rollers **207**, **208** and **209**. These folding rollers **207**, **208** and **209** have a straight shape.

The folding rollers **208** and **209** are severally pressed against the folding roller **207**. Namely, the folding rollers **207**, **208** and the folding rollers **207**, **209** are respectively in pairs. The folding rollers **207**, **208** which are paired will be referred to hereinafter as "paired folding rollers **207**, **208**" and the folding rollers **207**, **209** which are paired as "paired folding rollers **207**, **209**." The paired folding rollers **207**, **208** are disposed such that the nip part continues into the first conveying section **253**.

[Second conveying section **255**]

The second conveying section **255** is disposed between the downstream positions of the paired folding rollers **207**, **208** and the upstream positions of the paired folding rollers **207**, **209**. The second conveying section **255** comprises a second folding stopper **219** which regulates the second folding position of the sheet by contacting to the leading end

of the sheet, a solenoid (not shown) which switches the position of the second folding stopper **219** contacting to the sheet in conformity with the sheet size, a switching mechanism **218** which selectively guides the leading end of the sheet which has undergone the first folding by the paired folding rollers **207, 208** in the direction of the nip part of the paired folding rollers **207, 209** or in the direction of the second folding stopper **219**, and a solenoid (not shown) which rotates the switching device **218**.

[Discharging section **256**]

The discharging section **256** is disposed on the downstream side of the paired folding rollers **207, 209** and is possessed of discharging rollers **203** and **204**. The roller **203** constitutes itself one of the conveying rollers **202, 203**.

[Mechanism of restoring from jam]

The mechanism of restoring from a sheet jam which occurs in the folding section **254** of the folding device **200** will be described with reference to FIG. **4**.

The folding rollers **207, 208** and **209** in the folding section **254** are added with relatively high pressing force because they are required to fold the sheet strongly. The pressing force, for example, is 10 kg per roller. When the sheet happens to be wrapped fast around any of the folding rollers **207, 208** and **209**, it is a very difficult work to remove the stuck sheet, or solving the jam.

The folding device **200** of the present embodiment, therefore, releases either of the two folding rollers **208, 209** from being pressed against the folding roller **207** and opens the folding section **254** in order to improve the operational efficiency of restoring from the jam in the vicinities of the folding rollers **207, 208** and **209**. This construction will be described below.

An open unit **222** is formed by integrally retaining the second conveying section **255**, the single folding roller **209** and a guide **261** of the discharging section **256**. This open unit **222** is supported as freely rotated around a fulcrum **262** provided on a frame of the folding device **200**.

Further, a lock lever **220** constructed to encircle the periphery of the remotest section of the open unit **222** from the fulcrum **262** (as the upper end of the diagram) is supported as freely rotated around a fulcrum **263** provided on the frame. Lock shafts **227** are provided one each in the front and rear portions of the lock lever **220** extending in the direction perpendicular to the face of the sheet bearing an image. When the open unit **222** is closed, the lock shafts **227** are severally engaged with recess **22***a* formed in the open unit **222** and the open unit **222** is infallibly locked to the folding device **200**.

The lock lever **220** and the open unit **222** are connected through a link device **221**. The link device **221** enables the open unit **222** to be retained and rotated as synchronized with the rotation of the lock lever **220** and can preclude the fall of the open unit **222** during the relief of the lock.

[Detailed construction of first folding stopper]

As illustrated in FIG. **5**A, FIG. **5**B and FIG. **6**, the first folding stoppers **215, 216, 217** and **223** as devices for regulating the leading end of the sheet, the cams **211, 212** and **213**, the stepping motor **210**, and the cam shaft **224** are integrally held by a stopper unit frame **228**.

Excepting the stopper **223** disposed on the most downstream side in the conveying direction of the sheet, the first folding stoppers **215, 216** and **217** are constructed as freely rotated around respective fulcrums provided on the stopper unit frame **228**. The first folding stopper **223** is fixed to the stopper unit frame **228** and retained as constantly projected into the sheet conveying path.

The first folding stoppers **215, 216** and **217** are driven to move into and out of the sheet conveying path by the rotation of the cams **211, 212** and **213** and the cam shaft **224** which are disposed on the lower side of the frame **228**. The cams **211, 212** and **213** are attached in different angles to the cam shaft **224**. The first stoppers **215, 216** and **217** move severally into and out of the sheet conveying path while the cam shaft **224** produces one complete rotation. The stepping motor **210** rotationally drives the cam shaft **224**. One of the first folding stoppers **215, 216** and **217** is moved into and out of the sheet conveying path by actuating the stepping motor **210** in a desired angle proper for the folding mode or the sheet size.

The cam shaft **224** is provided with a light stop or gobo **231**. The gobo **231** is moved into and out of the detecting area of a home position sensor **230** in consequence of the rotation of the cam shaft **224**. The position at which the home position sensor **230** detects the gobo **231** is the home position for the cam shaft **224**. At the home position, all the first folding stoppers **215, 216** and **217** that are capable of moving into and out of the sheet conveying path are not in a projecting state except the first folding stopper **223**.

The first folding stopper **217** is designed to have the function of regulating two kinds of folding positions. To be specific, it is approximately shaped like a letter U having the opposite ends projected toward the upstream side in the conveying direction of the sheet as clearly shown in FIG. **6**. This shape is applicable only when the position for regulating the leading end of a sheet of a small width relative to the orthogonal direction falls on the downstream side in the conveying direction from the position for regulating the leading end of a sheet of a large width. Naturally, in this case, the stopper for the sheet of a large width must be disposed on the outer side along the orthogonal direction than the stopper for the sheet of a small width. In other words, the first folding stopper **217** is required to form, at the upstream position in the conveying direction, a notch of a width larger than the width of that of the two kinds of sheet which has a smaller width. The edges of the notch, or the edge located on the upstream side in the conveying direction and the edge located on the bottom, function as stoppers which contact to the leading edges of the two different kinds of sheet, respectively

In the illustrated embodiment, the first folding stopper **217** is constructed by integrating stoppers **217***a* disposed on the opposite outer sides used in double-folding of an A3 with a stopper **217***b* disposed on the further downstream side than the stopper **217***a* and used in Z-folding of a B4 sheet.

The anti-deviation device **226** is mounted where the first folding stoppers **215, 216, 217** and **223** contact to the leading end of the sheet as illustrated in FIG. **7**. The anti-deviation device **226** is provided for the purpose of precluding the inconvenience that the leading end of the sheet slides laterally on the contacting face of a stopper and induces deviation of the folding position. This fact explains why the anti-deviation device **226** is made of an elastic material with a high surface friction coefficient and a low hardness. The anti-deviation device **226** is also effective in abating the noise which is made when the leading end of the sheet contacts to the stopper.

The advantages of the construction are as follows.

Firstly, the deviation of positions occurring when the leading end of a sheet is regulated is slight. Because the devices for regulating the leading end of a sheet, or stoppers **215, 216, 217** and **223** are disposed one each at the plurality of positions used or required for regulating the leading end of a sheet.

Secondly, one motor **210** suffices as a drive source. Because the plurality of devices for regulating the leading end of a sheet can be actuated by the single cam shaft.

Thirdly, the components for actuation can be simplified.

Because a device for regulating the leading end of a sheet, or stopper **217** has the function of regulating the leading ends of two kinds of sheet and a device for regulating the leading end of a sheet on the most downstream side, or stopper **223** has a stationary structure. Namely, the function of regulating the leading end of a sheet can be accomplished with high accuracy by means of simple and inexpensive construction.

It is, when necessary, allowable to divide the drive system into two and add the cam shafts, etc. though one cam shaft and one motor suffice to actuate the plurality of devices for regulating the leading end of a sheet.

[Operation of various folding modes]

The folding device **200** has the three folding modes, (1) Z-folding, (2) double-folding, and (3) creasing. When the folding mode is inputted through a control panel provided in the copying machine **10**, the folding device **200** is controlled in the inputted mode.

(1) Z-Folding Mode

FIG. **8** is a cross section illustrating the state of the folding device **200** under the A3 Z-folding mode. In the diagram, the states which the sheet P assume at different points of time are simultaneously indicated in the folding device **200** as well as in FIGS. **9** and **10**.

The term "Z-folding mode" refers to a mode of folding a sheet of a large size (A3 or B4) in a cross section like a letter Z, or in the sheet in a size approximately one half of the original length of the sheet along the conveying direction The sheet P outputted from the sheet output section **10***b* of the copying machine **10** is conveyed in the "longitudinal" direction to the switch claw **201**, with the image-formed face held on the upper side. The sheet P is fed into the folding device **200** by the rotation of the switch claw **201** and then nipped by the conveying rollers **202**, **203**. The sheet P is further conveyed to the adjusting section **252** wherein the leading end of the sheet is corrected by removal of a deviation. Thereafter, the sheet P is conveyed toward the first folding stoppers **215**, **216**, **217** and **223**.

Immediately after the command of copy start is inputted, the stepping motor **210** is rotated by a fixed number of steps proper for the sheet size and the folding mode to set the position of the first folding stopper **215** **216** or **217** (projecting position or retracting position). All three first folding stoppers **215**, **216** and **217** are retracted and the fixed first folding stopper **223** alone is projected when the sheet has the size of A3 and is in the longitudinal direction under the Z-folding mode as illustrated in the diagram. The first folding stopper **217** is moved to the projected position when the sheet has the size of B4 and is in the longitudinal direction.

After the leading end of the sheet has contacted to the first folding stopper **223**, the conveyance of the sheet is further continued. As a result, the sheet forms a loop in the neighborhood of the nip of the paired folding rollers **207**, **208** and the loop is finally gripped by the nip of the paired folding rollers **207**, **208**. Consequently, the first folding is effected on the sheet.

A guide **264** near the nip of the paired folding rollers **207**, **208** is naturally constructed in a shape such that the loop in the sheet P is infallibly formed steadily as directed to the nip of the paired folding rollers **207**, **208**.

The first folding position is separated by approximately ¾ of the total length of the sheet in a given sheet size from the edge of the sheet, or the leading end side in entering the folding device **200**. In this specification, for the sake of convenience of description, the first fold will be defined as "three-quarter (¾) fold." The first fold at the position separated by approximately ¼ of the total length of the sheet from the edge of the sheet will be defined as "one-quarter (¼) fold."

In response to the command "Z-folding" from the copying machine **10**, the switching device **218** is moved to the position for leading the sheet P in the direction of the second folding stopper **219**. The leading end of the sheet P conveyed by the paired folding rollers **207**, **208** contacts to the second folding stopper **219** which has been switched in accordance with the sheet size.

When the conveyance of the sheet P is continued by the paired folding rollers **207**, **208** after the leading end has contacted to the second stopper **291**, the sheet P forms a loop near the nip of the paired folding rollers **207**, **209**. This loop is finally gripped by the nip of the paired folding rollers **207**, **209**. The second folding position is at a distance of approximately ½ of the total length of the sheet.

Here again, a guide **265** near the nip of the paired folding rollers **207**, **209** is naturally constructed in a shape such that the loop in the sheet P is infallibly formed steadily as directed to the nip of the paired folding rollers **207**, **209**.

The sheet P on which the Z-folding has been completed by the second folding is conveyed toward the discharging section **256** by the paired folding rollers **207**, **209** and discharged from the folding device **200** by the discharging rollers **203**, **204**.

The Z-folding mode can do a so-called mixed working, i.e. an additional-working on a mixture of folded sheets and unfolded sheets. To be specific, Z-folding mode can achieve the mixed working of **A3** Z-folding in the longitudinal direction and unfolded **A4** sheets in the lateral direction or the mixed working of **B4** Z-folding in the longitudinal direction and unfolded **B5** sheets in the lateral direction.

Under the mixed mode, sheets for folding can be fed at a standard interval into the finisher **100** when the sheets follow sheets for no folding into the finisher **100**. Conversely, feeding of the sheets for no folding at the standard interval into the finisher **100** possibly causes such inconveniences as disruption of the order of pages or the contact between the sheets when the sheets follows sheets for folding into the finisher **100**. The present embodiment, therefore, precludes in the latter case, the occurrence of such inconveniences as the disruption of the order of pages by loading a weight on the conveyance of the sheets for no folding and preventing these sheets from entering the finisher **100** until the folded sheets are discharged from the folding device **200**.

In consideration of the appearance of the product of the mixed working, the second crease or fold is preferably prevented from jutting out of the unfolded sheets. For this reason, the second folding position preferably deviates slightly from the ½ position of the total length of the sheet toward the edge of the sheet as the leading end side in entering the folding device **200**.

(2) Double Folding Mode

FIG. **9** is a cross section illustrating the state of the folding device **200** under the **A3** double-folding mode.

The term "double-folding mode" refers to the mode of folding a sheet in two or the central section.

The sheet P discharged from the sheet output section **10***b* of the copying machine **10** undergoes the same process as under the Z-folding mode and conveyed toward the first folding stoppers **215**, **216**, **217** and **223**.

Likewise under the double-folding mode, the stepping motor **210** is controlled to move only the first folding stopper **217** to the projecting position when the sheet has the size of **A3** and is in the longitudinal direction, as illustrated in the

diagram. The first folding stopper **216** is only moved to the projecting position when the sheet has the size of B4 and is in the longitudinal direction. The first folding stopper **215** is only moved to the projecting position when the sheet has the size of A4 and is in the longitudinal direction. The sheet P, after undergoing the same process as under the Z-folding mode, is gripped by the nip of the paired folding rollers **207**, **208** and then given the first folding.

In response to the command "double-folding" from the copying machine **10**, the switching device **218** is moved to the position for guiding the sheet P toward the nip of the paired folding rollers **207**, **209**. Then, the sheet P conveyed by the paired folding rollers **207**, **208** is gripped on the crease by the nip of the paired folding rollers **207**, **209** and conveyed per se to the paired discharging rollers **203**, **204** and discharged from the folding device **200**.

(3) Creasing Mode

FIG. **10** is a cross section illustrating the state of the folding device **200** under the creasing mode.

The term "creasing mode" refers to the mode of preparatorily creasing the central section of sheet for stapling the central crease of the sheaf like a weekly magazine.

The sheet P discharged from the sheet output section **10***b* of the copying machine **10** is conveyed toward the first folding stoppers **215**, **216**, **217** and **223**, similarly to the Z-folding mode or the double-folding mode.

The folding position under the creasing mode is identical with that under the double-folding mode. The motions of the first folding stoppers **215**, **216** and **217** are controlled in the same manner as under the double-folding mode. And the sheet P is gripped by the nip of the paired folding rollers **207**, **208** and given the first folding.

In response to the command "creasing mode" from the copying machine **10**, the switching device **218** is moved to the position for guiding the sheet P toward the second folding stopper **219**. The sheet P which has undergone the first folding is conveyed by the paired folding rollers **207**, **208** toward the second folding stopper **219**.

The driving direction of the rollers **202**, **205** and **207** in the folding device **200** is switched from the normal rotation (the direction of the arrow a in the diagram) to the reverse rotation (the direction of the arrow b in the diagram) after the elapse of the period of the time t2 [second] which follows the detection of the trailing edge of the sheet P having undergone the first folding by the sheet sensor **225** in the feed channel section **251**. The term "t2" refers to the length of time satisfying the following condition:

$$(y/V)>t2>(x/V)$$

in which V stands for the rate of conveyance of a sheet, x for the distance between the sheet sensor **225** and the lower edge of the switch claw **201**, and y for the distance between the leading end of the sheet and the second folding stopper **219** after the detection of the trailing end of the sheet and the completion of the first folding.

The crease formed in the central section of the sheet P is released from the paired folding rollers **207**, **208** in consequence of the reverse rotation of the rollers **202**, **205** and **207**. The edge, which has been the trailing edge during the feed of the sheet into the folding device **200**, is now the leading edge. And the sheet is led to the switch claw **201** held in the same state as during the feed of the sheet, and passed through the path indicated by the arrow W, and discharged from the folding device **200**. In this manner, the sheet P with the central crease can be conveyed in an opened posture toward the downstream side.

Incidentally, all the three folding modes are invariably accepted only when the sheet has a length of not less than twice the length of the sheet of the smallest size that is available for conveyance.

[Turn-back of sheet during the folding]

A turn-back mechanism **20**, which turns a sheet with a copied image upside down, is installed near the sheet output section **10***b* of the copying machine **10**. This turn-back mechanism **20** comprises a path for switchback conveyance of a sheet and a pair of reversible rollers provided in the path. The turn-back mechanism promotes compaction of the finisher and reduction in the cost. The arrangement of the turn-back mechanism **20** does not need to be limited to the vicinity of the sheet output section **10***b* of the copying machine **10**. This mechanism **20** may be disposed closely to the feed channel section **150** of the finisher **100** instead.

The copying machine **10** further comprises three paths **21**, **22** and **23** used as selectively switched. The first path **21** is applied to discharge the sheet turned by the turn-back mechanism **20** from the sheet output section **10***b*. The second path **22** is applied to rotate the sheet turned by the turn-back mechanism **20** within the copying machine **10** for two-sided copies or copying an image on the side opposite to the side with the copied image. The third path is applied to directly discharge the sheet from the sheet output section **10***b* without passing the sheet through the turn-back mechanism.

The copying machine **10**, based on the operating mode set by the user and the size of the sheet selected for copying, judges whether or not the sheet for copying is subsequently folded and inputs the information resulting from this judgment to the finisher **100**.

FIG. **11** is a flow chart illustrating the process for setting a sheet conveying path.

When the copy mode is not a two-sides copies mode ("N" at Step S**11**) and the judgment is "sheet for folding" ("Y" at Step S**12**), the copying machine **10** switches the conveying path to the third path **23** (Step S**13**). Then, the sheet is discharged from the sheet output section **10***b* without passing through the turn-back mechanism. In contrast, when the judgment is "sheet for no folding" ("N" at Step S**12**), the copying machine **10** switches the path to the first path **21**. Then, the sheet is passed through the turn-back mechanism **20** and discharged in a reversed state from the sheet output section **10***b* (Step S**14**). The finisher **100**, based on the information inputted from the copying machine **10**, controls the rotation of the switch claw **201** disposed on the upstream side of the folding device **200** and the positions of the first and second folding stoppers **215**, **216**, **217**, **223** and **219** in conformity to the relevant folding mode.

When the copy mode is a two-sides copies mode ("Y" at Step S**11**), the conveying path is temporarily switched to the second path **22** ("N" at Step S**15**, S**16**) after the first copy is completed on one side. After the second copy is completed on the other side ("Y" at Step S**15**), the operation described above is executed, depending on the result of the judgment whether or not the sheet folding is necessary.

[Retracting of first fold stopper during restoring from jam]

The sheet folding in the folding device **200** is achieved by contacting to the leading end of a sheet to the first and second folding stoppers **215**, **216**, **217** **223** and **219**, and forming a loop halfway in the entire length of the sheet, and gripping the loop with the folding rollers **207**, **208** and **209**. The plurality of first folding stoppers **215**, **216** and **217** disposed along the conveying direction of the sheet are moved in and out by the cams **211**, **212** and **213** connected to the stepping motor **210** as the drive source and can be retracted outside the sheet conveying path. The stepping motor **210**, which actuates the cam shaft **224**, is rotated by

an angle proportionate to the number of received pulses. The forward and backward motions of the first folding stoppers **215**, **216** and **217** are controlled in terms of the angle of rotation of the cam shaft **224** in response to the number of pulses inputted to the stepping motor **210** based on a home position at which the gobo **231** provided on the cam shaft **224** is detected by the home position sensor **230**. The home position is defined as where all the first folding stoppers **215**, **216** and **217** capable of forward and backward motions are retracted outside the conveying path.

In the folding device **200** of this embodiment, the ¾ fold as the first fold of Z-folding mode is done by setting the first folding stoppers at the positions separated from the paired folding rollers **207 208** by a distance equivalent to the length of ¾ of the sheet size. Accordingly, the first folding stopper **223** is only fixed at the position separated from the paired folding rollers **207**, **208** by a distance equivalent to the length of ¾ of the largest sheet size (**A3** in the longitudinal direction) in all the sheet sizes (**A3** in the longitudinal direction and **B4** in the longitudinal direction) that are capable of Z-folding.

FIG. **12** is a flow chart illustrating the retracting process of the first folding stopper during the restoration from a sheet jam.

When a jam occurs in the folding device **200** ("Y" at Step **S21**), it is judged whether or not the first folding stoppers **215**, **216** and **217** are at the home position, based on the signal from the home position sensor **230** (**S22**).

When the first folding stoppers **215**, **216** and **217** are not at the home position ("N" at Step **S22**), the stepping motor **210** is kept in operation until the stoppers **215**, **216** and **217** return to the home position, namely until the gobo **231** provided on the cam shaft **224** is detected by the home position sensor **230** (**S22**, **S23** and **S24**). Namely, the first folding stoppers **215**, **216** and **217** return to the home position and then the fact that the sheet jam has occurred in the folding device **200** is outputted on the control panel of the copying machine **10**.

When the completion of the restoration from a sheet jam is detected, the first folding stoppers **215**, **216** and **217**, which have been retracted to the home position, are moved to the position which was assumed when the jam actually occurred.

In brief, the first folding stopper is retracted outside the sheet conveying path and a space large enough for the user to insert his hand to the vicinity of the sheet in the jam is formed when the jam of sheet occurs. Consequently, the user can easily insert his hand and remove the jammed sheet. And there is no possibility that the user will accidentally touch and move the first folding stopper during the restoration from the jam and the first folding stopper keeps its accurate position. Further, no addition of any special mechanism is required and no possibility of the user accidentally touching the first folding stopper can set the strength of the finisher at the level of an irreducible minimum. Consequently, the finisher enjoys simplicity of construction and low cost.

<<Punching unit **300**>>

FIG. **13** and FIG. **14** are respectively a perspective view and a side view illustrating the punching device **300**.

The punching device **300** comprises a punch blade **303**, a punch die **307** which makes a hole in cooperation with the punch blade **303**, a drive cam **301** which moves the punch blade **303** forward and backward by contacting to the punch blade **303**, and a resist roller **308** (FIG. **2**) which fixes the punching position. The punch die **307** is mounted on the lower side of a base plate **306** as separated by a certain gap S.

The drive cam **301** is left standing at a certain stop position while the punch is not in use. A drive shaft **302** on which the drive cam **301** is mounted is connected to a motor **304** through a solenoid clutch **305**. The drive cam **301** produces one rotation and returns to the stop position and stops when the solenoid clutch **305** is turned on and the motor **304** rotates the drive shaft **302**. The punch blade **303** produces one reciprocation in consequence of one rotation by the drive cam **301**.

The punch die **307** has a hole with an inside diameter nearly equal to the outside diameter of the punch blade **303**. The punch blade **303** fits into the hole in the punch die **307** when the punch blade **303** is moved by the largest stroke at least from the stop position. And a punched hole is formed in a sheet by gripping or inserting the sheet in the gap S between the punch blade **303** and the punch die **307** and reciprocating the punch blade **303** once.

The punch blade **303**, as illustrated in FIG. **2**, is disposed on the downstream side of the resist roller **308** along the conveying direction. The distance between the punch blade **303** and the nip of the resist roller **308** is set so as to equal a size required to intervene between the punched hole formed in the sheet and the edge of the sheet. The sheet sensor **102** formed of a photosensor, for example, is provided on the upstream side of the resist roller **308**. The sheet discharged from the folding device **200** or brought in without passing through the folding device **200** is caused to change the destination by the switch claw **103** on the downstream side of the punching device **300** and is conveyed by the conveying roller **104** or the conveying roller **121**. These two conveying rollers **104**, **121** may stop at an arbitrary timing through a solenoid clutch.

The conveying rollers **104**, **121** are stopped in a state such that the trailing end of the sheet remains in a slight amount on the upstream side from the resist roller **308** when the sheet sensor **102** detects the trailing end of the sheet. The resist roller **308** is continuously rotated even after the conveying rollers **104**, **121** have been stopped. As a result, the sheet continues to stand at rest in the state such that the trailing end remains in the nip of the resist roller **308**. In the sheet left standing in the state, the punched hole is formed by one reciprocating motion of the punch blade **303**. Thereafter, the solenoid clutch of the conveying rollers **104**, **121** is again turned on and the punched sheet is conveyed further to the downstream side.

In conclusion, the punching device **300** is operated as described above to form a punched hole separated by a fixed interval from the trailing end of the sheet.

<<Additional work tray unit **400**>>

FIG. **15** is a cross section illustrating the construction of the additional-work tray unit **400** and the stapler **500** disposed on the downstream side and FIG. **16** and FIG. **17** are respectively a lateral section and a partially cutaway bottom view illustrating an additional-work tray **401** of the additional-work tray unit **400**.

For the sake of convenience of the description, the alignment along the conveying direction from the additional-work tray **401** to the stapler **500** (FD-direction) will be referred to as "FD-alignment" and the alignment along the width direction of conveying sheet, i.e. the orthogonal direction (CD-direction), as "CD-alignment" hereinafter.

The additional-work tray unit **400** comprises the additional-work tray **401** which temporarily stores, in a face-down state, the sheet which is reversed upside down in the upstream section and then discharged by the discharging roller **113**, a leading end stopper **409** which is disposed in the sheet discharging outlet **401***a* of the additional-work tray

401 and effects the FD-alignment of the sheet, a pair of lateral aligning plates **402** which effects the CD-alignment of the sheet discharged by the discharging roller **113**, a trailing end stopper **403** which stabilizes the FD-alignment done with the leading end stopper **409** by contacting to the leading end of the sheet discharged by the discharging roller **113**, and the first sheaf-conveying rollers **114**, **115** which conveys a certain number of sheets stored in the additional-work tray **401** as one sheaf to the stapler **500**.

The additional-work tray **401** corresponds to the receiving tray unit for storing sheets. The leading end stopper **409** as the first regulating device is disposed as projected from the additional-work tray **401** and contacts to one end face of the sheaf stacked in the additional work tray **401**. The trailing end stopper **403** as the second regulating device is disposed as projected from the additional-work tray **401**, and contacts to the leading end of the sheet being conveyed to the additional-work tray **401** and pushes the other end face of the sheaf inward until the one end face is aligned with the leading end stopper **409**.

The additional-work tray **401** is set up such that the sheet-discharging outlet **401***a* is inclined downward by a certain angle. The pair of lateral aligning plates **402** is disposed such that they are freely moved symmetrically along the CD-direction. The pair of lateral aligning plates will be occasionally-referred to hereinafter otherwise as "paired lateral aligning plates." The trailing end stopper **403** is disposed so as to move along the FD-direction freely. The CD-alignment is effected each time that the additional-work tray **401** receives a sheet. Additionally, the FD-alignment is effected each time that the additional-work tray **401** receives a sheet or a certain number of sheets. The first sheaf-conveying rollers **114**, **115** constitute a pair of the lower roller **114** and the upper roller **115**. The upper roller **115** can move substantially in the vertical direction to press the lower roller **114** or depart from the lower roller **114**.

[Paired lateral aligning plates **402**]

The paired lateral aligning plates **402**, as illustrated in FIG. **15** and FIG. **16**, are composed of plates having a height (L1) greater than the largest height of the sheaf that can be stored on the additional-work tray **401**. The paired lateral aligning plates **402** are severally mounted on a pair of racks **420** provided on the reverse side of the additional-work tray **401** along the CD-direction. The paired racks **420** are mounted as opposed to each other across a gear **421** which is rotatably driven by a stepping motor **408**. The rotation of the gear **421** causes the paired lateral aligning plates **402** to move symmetrically along the CD-direction. To be specific, the paired lateral aligning plates **402** synchronously move toward each other during the normal rotation of the stepping motor **408** and synchronously move away from each other during the reverse rotation of the stepping motor **408**.

The paired lateral aligning plates **402** have two waiting positions, i.e. a first waiting position and a second waiting position. The first waiting position is a place occupied before the discharging roller **113** discharges the sheet. The second waiting position, as altered by the size of the sheet to be discharged, occupies a slightly wider area than the size of the sheet and is a place for awaiting the discharge of the sheet by the discharging roller **113**. The paired lateral aligning plates **402** are freely moved between the three positions, i.e. the first waiting position, the second waiting position, and the position for the CD-alignment of the sheet discharged by the discharging roller **113**.

A plurality of sensors **410** for positioning the paired lateral aligning plates **402** are provided on the lower face of the additional-work tray **401**. The gobos, or stops for intercepting the light from the sensors **410** are integrally mounted on the paired lateral aligning plates **402**. Positioning of the first and second waiting positions are based on that the gobos intercept the light from the sensors **410**. The positioning of the paired lateral aligning plates **402** for the alignment is done by controlling the number of pulses inputted the stepping motor **408** to actuate the gear **421**.

[Leading end stopper **409**]

The leading end stopper **409**, as illustrated in FIG. **15** and FIG. **17**, is roughly shaped like a letter L, and is composed of a bottom plate **409***a* and a blocking plate **409***b* raised from the leading end of the bottom plate **409***a*. The leading end stopper **409** is so mounted on the lower face of the additional-work tray **401** to freely rotate about a fulcrum **430** provided on the bottom plate **409***a*. The leading end stopper **409** is urged by the elastic force of a spring **431** to contact to the lower face of the additional-work tray **401**. The blocking plate **409***b* of the leading end stopper **409** forms a base plane when the FD-alignment is effected on the sheet to be stored in the additional-work tray **401**. The blocking plate **409***b* of the leading end stopper **409** is moved downward as indicated by an alternate two-dot chain line in FIG. **15**, by actuating a solenoid to pull a link arm (not shown) pivotally supported on a rotary fulcrum **430**. It results in opening the sheet-discharging outlet **401***a*for feeding a sheaf to the stapler **500**.

[Trailing end stopper **403**]

The trailing end stopper **403**, as illustrated in FIG. **15**, is disposed on the side such that the crease of a sheet exists on the additional-work tray **401**. The trailing end stopper **403** comprises a plate **412**, a sponge **411** attached to one face of the plate **412** to which the sheet contacts, and a framer **413** supporting the plate **412**. Roughly the upper half of the plate **412** is rounded, or radius-shaped by being projected as slightly curved from the direction perpendicular to the upper face of the additional-work tray **401** toward the leading stopper **409** located on the sheet discharging outlet **401***a*.

The plate **412** of the trailing end stopper **403** with the rounded shape produces the following advantages. The trailing end of the sheet along the conveying direction from the additional-work tray **401** to the stapler **500** (corresponding to the leading end of the sheet being discharged from the discharging roller **113**) always contacts steadily to the plate **412** of the trailing end stopper **403** without reference to the number of sheets stacked on the additional-work tray **401**, the size of the sheet, or the presence or absence of the folding. In consequence of this contact, the sheet is repelled in the direction opposite the discharging direction and the leading end of the sheet along the conveying direction infallibly contacts to the leading end stopper **409** and the FD-alignment is further ensured. The Z-folding sheet, owing to the crease, has the trailing end along the conveying direction in a slightly lifted state. However, the sheaf including Z-folding sheets can be uniformly pushed in and brought into contact with the leading end stopper **409** by using the plate **412** having the radius-shaped upper part. Thus, the additional-work tray unit **400** can infallibly eliminate the deviation in the conveying direction possibly produced in the sheaf including Z-folding sheets during the conveyance to the-stapler **500**.

The framer **413** of the trailing end stopper **403**, as additionally illustrated in FIG. **17**, is engaged with a spiral shaft **404** which is installed as extended along the conveying direction at the center of the lower face of the additional-work tray **401**. This spiral shaft **404** is connected to a motor **406** as a DC motor through a transmission device **435** as a gear train. The trailing end stopper **403** is moved forward or

backward by a necessary distance along the conveying direction by actuating the motor **406** properly in the normal or reverse direction to rotate the spiral shaft **404**.

A home position sensor **405** formed of a photosensor, for example, is mounted on a casing **440** supporting the spiral shaft **404** as illustrated in FIG. **15**. And a gobo or a stop (not shown) for intercepting the light from the sensor **405** is mounted on the framer **413** of the trailing end stopper **403**. The trailing end stopper **403** is stopped at a certain home position on the additional-work tray **401** based on the detection of this stop by the sensor **405**. A pulse disc sensor **407** is so mounted on the drive shaft of the motor **406** as to stop the trailing end stopper **403** highly accurately at a necessary position in response to the signal of a conventional pulse disc sensor **432** (FIG. **17**).

[Sheet alignment in additional-work-tray unit **400**]

FIG. **18**A–FIG. **18**C are explanatory diagrams illustrating steps for the sheet alignment in the additional-work tray unit **400**, FIG. **18**D is an explanatory diagram illustrating steps for the conveyance of a sheaf of stacked and aligned sheets to the stapler **500**, FIG. **19** is a diagram illustrating the states of various staple modes, FIG. **20** is a flow chart illustrating the control of the trailing end stopper **403**, and FIG. **21** is a flow chart illustrating the operation of the first sheaf-conveying rollers **114**,**115** during the sheet alignment.

Now, steps for the sheet alignment in the additional-work tray unit **400** will be described as divided into (1) a version in the absence of Z-folding sheets and (2) a version in the presence of Z-folding sheets.

(1) Absence of Z-Folding Sheet

When the sheets are temporarily stacked on the additional-work tray **401** for stapling in the absence of Z-folding sheet, the leading ends of the sheets discharged by the discharging roller **113** are caused to contact or collide against the trailing end stopper **403**. Then, the leading ends of the sheets are caused to contact to the leading end stopper **409** by virtue of the repelling force arising from the collision and the weight of the sheets. The series of motions effect the FD-alignment. The movement of the paired lateral aligning plates **402** in the CD-direction effects the CD-alignment. A discharged sheet sensor **112** which detects the trailing end of a sheet for judging the discharge of the sheet from a first conveying path **441** toward the additional-work tray **401** is disposed in the upstream vicinity of the discharging roller **113**.

To be more specific, the trailing end stopper **403** moves to and stops at the second stop position keeping a stated distance from the end face of the sheet, depending on the presence or absence of sheet folding, the size of sheet, and the mode of sheet folding (**S32**), as shown in the flow chart of FIG. **20**. Besides, the conveying length to the second stop position is calculated at Step **S31** in accordance with the formula, Lm (moving length) =Lt (length of the additional-work tray **401**)–Ls (size of sheet)–α (certain distance). The term "Ls (size of sheet)" refers to the size of the sheet measured when the sheet is fed into the additional-work tray **401**. Thus, Ls (size of sheet) is the size of the folded sheet when the sheet is folded. The term""α (certain distance)x varies, depending on the presence or absence of folding.

The leading end of the sheet being discharged from the discharging roller **113** contacts to the trailing end stopper **403** kept at the second stop position or the calculated position. Then, the sheet is repelled toward the leading end stopper **409** and quickly dropped onto the upper face of the additional-work tray **401** or on the uppermost of the stored sheets. The FD-alignment, therefore, can be optimally and expeditiously carried out even when the sheets are conveyed and discharged at a small interval. Moreover, the timing for the subsequent CD-alignment can be quickly set. The temporary storage of sheets in the additional-work tray **401** is completed early and the series of additional-workings can be fulfilled expeditiously. Thus, the finisher is improved in productivity.

In the absence of a Z-folding sheet, the trailing end stopper **403** is retained at the second stop position until the storage of sheets for one job is completed and then is returned to the home position (**S33**, **S34** and **S35**).

The trailing end stopper **403** may be controlled to move to a position at which the distance to the leading-end stopper **409** equals the size of the discharged sheet and contact to the trailing end of the sheaf before the return of the trailing end stopper **403** to the home position.

Incidentally, the sheets temporarily stored in the additional-work tray **401** have been slightly curled under the influences of heat and pressure exerted by the formation of images. There is the possibility-that the leading ends of sheets partly ride aslant on the blocking plate **409***b* of the leading end stopper **409**. When the sheets in the above state are aligned without correction and stapled, the stapled sheaf suffers from poor appearance because the mismatch in the FD-direction of the sheaf is in existence.

For solving the problem, the additional-work tray unit **400** waits for a certain length of time to elapse after the sensor **112** has detected the discharge of the sheet and then lowers, toward the lower roller **114**, the upper roller **115** constructed to be freely moved toward and away from the lower roller **114** under the FD-alignment as illustrated in FIG. **18**B and FIG. **18**C. The certain length of time is sufficient to be spent by the leading end of the discharged sheet in coming into contact to the leading end stopper **409**. By lowering the upper roller **115** once after the discharge of the sheets, those sheets ridden aslant on the blocking plate **409***b* of the leading end stopper **409** are dropped onto the additional-work tray **401** and are aligned by eliminating the deviation of the FD-direction. Namely, the FD-alignment by the leading end stopper **409** is infallibly achieved. Then, the upper roller **115** is moved upward before the subsequent sheet comes and is prevented from contacting to the subsequent sheet.

Incidentally, the upper roller **115** as the first sheaf-conveying roller is arranged to produce no rotation at least during the descent. Owing to this arrangement, the upper roller **115** avoids compelling the sheet to incur such inconveniences as production of wrinkles under a pressure from the roller. This construction will be described specifically herein below.

In respect of receiving the first sheet for storage, the lower roller **114** as the first sheaf-conveying roller protrudes upward from the stacking base of the additional-work tray **401** as illustrated in FIG. **18**A. There is the possibility that the leading end of the first sheet entering the additional-work tray **401** contacts to the lower roller **114** and gets stuck in this portion.

In consideration of the point, the lower roller **114** continues to advance only the first sheet by producing several rotations even after the sensor **112** has detected the discharge of the sheet for enabling the leading end of the sheet to contact to the leading end stopper **409** infallibly. Specifically as shown in the flow chart of FIG. **21**, the lower roller **114** is actuated to produce n rotations when the absence of a sheet on the additional-work tray **401** is discerned and the sensor **112** has detected the trailing end of the sheet (**S41**–**S43**). The first sheet, therefore, can be aligned properly. With respect to the second and following sheets, only the motion of the upper roller **115** toward and away from the

lower roller **114** is effected because sheets are already present on the additional-work tray **401** (S44).

Incidentally, if the operation of lowering the upper roller **115** into forced contact to the upper roller **114** is effected additionally on the first sheet, the first sheet will be pressed strongly against the leading end stopper **409** and possibly incur inconveniences. The motion of the upper roller **115** toward and away from the upper roller **104** is carried out on the second and following sheets received in the additional-work tray **401**, or every sheets except the first sheet.

If the rotation of the lower roller **114** is continued during the receiving of second and following sheets, these sheets will be unduly advanced in consequence of a gradual increase in the cumulative weight of sheets stacked on the lower roller **114**. The-rotational operation of the lower roller **114**, therefore, is carried out exclusively when the first sheet is received into the additional-work tray **401** and is stopped during the receiving of the second and following sheets.

When the stacking, CD-alignment and the FD-alignment of a plurality of sheets on the additional-working tray **40** are completed, the upper roller **115** is moved downward and the first sheaf-conveying rollers **114, 115** which are now in a mutually pressed state nip the sheaf on the additional-work tray **401** as illustrated in FIG. **18**D. The leading end stopper **409** is further rotated to move the blocking plate **409***b* downward and to open the sheet discharging outlet **401***a* and a second conveying path **442** provided with the stapler **500**. The first sheaf-conveying rollers **114, 115** are then set rotating and the sheaf is passed through the sheet discharging outlet **401***a* and conveyed in the direction of the stapler **500**.

In the present embodiment, the first sheaf-conveying rollers **114, 115** can both convey the sheaf to the stapler **500** and align the sheet for eliminating a FD-direction deviation. This embodiment, therefore, simplifies or miniaturizes the finisher **100** as a whole and contributes also to lower the cost as compared with a finisher provided independently with a sheaf conveyance mechanism and an alignment mechanism.

The example using the first sheaf-conveying rollers **114, 115** as a device for conveying a sheaf is depicted in the drawing. The motion of the opposed rollers toward and away from each other for adjusting the FD-direction deviation can be applied to a sheaf conveying device which is composed of a chuck capable of nipping the sheaf with sliding.

(2) Presence of Z-Folding Sheet

The stapler has three staple modes, i.e. normal staple mode (FIG. **19**A), fold staple mode (FIG. **19**B), and mixed staple mode (FIG. **19**C), which are selectively adopted. The normal staple mode is a mode for stapling a sheaf solely of unfolded sheets, the fold staple mode is a mode for stapling a sheaf solely of folded sheets, and the mixed staple mode is a mode for stapling a sheaf of unfolded and folded sheets.

Without reference to the kind of staple mode, the folded and/or unfolded sheets are stacked on the additional-work tray **401** prior to the relevant stapling, subjected to the CD-alignment by the paired lateral aligning plates **402**, and then subjected to the FD-alignment performed jointly by the trailing end stopper **403** and the leading end stopper **409**.

Particularly, the trailing end stopper **403** can be freely moved to and stopped at a pertinent position in the FD direction. As shown in the flow-chart of FIG. **20**, the trailing end stopper **403** is moved to and stopped at the second stop position kept at a stated distance from the end face of the sheet, depending on the presence or absence of sheet folding, the size of sheet, and the mode of sheet folding for the purpose of effecting the FD-alignment perfectly (**S31** and **S32**).

The z-folding sheets which have a peculiar form such that there are three overlapping layers in one half of the length of sheet and one only one layer in the remaining half of the length, are stacked on the additional-work tray **401** such that the overlapping side is located on the side of the trailing end stopper **403**. In the mode involving a sheaf including Z-folding sheets, the sheets stacked on the additional-work tray **401** are not well balanced and have a possibility of partly protruding in the conveying direction.

The Z-folding constitutes itself a fold mode of folding a sheet (such as **A3** sheet) to a size (such as ΔL=about 3 mm) smaller than the size of an unfolded sheet (such as **A4** sheet) as illustrated in FIG. **22**A. In the FD-alignment of a sheaf including Z-folding sheets, perfect FD-alignment is not done by moving the trailing end stopper **403** in conformity with the sheets which are not Z-folding sheets.

In view of the factors, when the sheaf includes Z-folding sheets, the final sheet is received ("Y" at Step **S36** in FIG. **20**) and then the trailing end stopper **403** at the second stop position is moved to the position for pushing the Z-folding sheet into the leading end stopper **409** as illustrated also in FIG. **22**B and returned to the home position (**S37**). Therefore, the deviation of the sheets in the FD-direction can be eliminated even when the sheaf includes Z-folding sheets.

The Z-folding sheets assume a peculiar form. The sheaf of Z-folding sheets is not parallel to the stacking base of the additional-work tray **401** but is in an oblique state such that a section on the side of the trailing end stopper **403** is higher than a section on the side of the leading end stopper **409**. This oblique state grows as the number of Z-folding sheets included in the sheaf increases. The distance along the conveying direction between the trailing end of the sheets in the upper section of the sheaf and the leading end stopper **409** becomes short as compared with the distance between the trailing end of the sheets in the lower section of the sheaf and the leading end stopper **409**. Here, the sheets in the upper section of the sheaf are discharged in the final stacking stage and the sheets in the lower section of the sheaf are discharged in the initial stacking stage. In the state such the distance between the trailing end of sheet and the leading end stopper **409** varies in the upper section and the lower section of the sheaf, the sheets in the upper section of the sheaf will not be given a perfect FD-alignment when the trailing end stopper has a shape perpendicularly intersecting the stacking base of the trailing end stopper **401**.

In the present embodiment, roughly the upper half of the trailing end stopper **403** is rounded, or radius-shaped and inclined toward the sheet storing side. This construction enables the FD-alignment of the sheaf including Z-folding sheets to be effected uniformly and satisfactorily throughout the entire sheaf from the lower to the upper section.

After the CD-alignment and the FD-alignment are completed in the additional-work tray **401**, the sheaf is nipped by the first sheaf-conveying rollers **114, 115** and passed through the sheet discharging outlet **401***a* opened in consequence of the rotation of the leading end stopper **409** and then conveyed toward the stapler **500**.

Optionally, the trailing end stopper **403** may be controlled, at an interval of a suitable number of sheets, to move to a position at which a Z-folding sheet is pushed to the leading end stopper **409** and then returned to the second stop position before the final sheet has been received.

[Retracting of paired lateral aligning plates **402** and trailing end stopper **403** during restoration from sheet jam]

The paired lateral aligning plates **402** are located based on the pulses inputted to the stepping motor **408** and the signal

outputted from the sensor **410** which detects the paired lateral aligning plates **402** at the home position. The paired lateral aligning plates **402**, after discerning the size of a sheet for copying, move to a position separated slightly from the lateral end of the sheet and assume a waiting posture and effect the CD-alignment by making reciprocating motion each time one sheet is received for storage. The home position is separated by a minute length from the lateral end of a sheet stored in the additional-work tray **401**, which has the largest length in the CD-direction.

The trailing end stopper **403** is located in accordance with the pulses outputted from the pulse disc sensor **407** provided as a pulse generating device in the motor **406** and the signal outputted from the sensor **405** detecting the trailing end stopper **403** at the home position. The trailing end stopper **403**, after discerning the size of sheet for copying and the mode of sheet folding, produces a motion proper for the size of a sheet received into the additional-work tray **401**. The home position is separated by a minute length from the trailing end of a sheet stored in the additional-work tray **401**, which has the largest length in the FD-direction.

When a jam occurs inside the additional-work tray unit **400**, first the paired lateral aligning plates **402** and the trailing end stopper **403** are returned to their respective home positions and then the fact that the jam has occurred in the additional-work tray unit **400** is outputted on the control panel on the copying machine **10**.

After the completion of the restoration from the jam is detected, the paired lateral aligning plates **402** and the trailing end stopper **403** are both moved to the positions which they occupied when the jam occurred.

<<Stapler **500**>>

[Construction of stapler **500**]

FIG. **23** is a structural diagram illustrating the stapler **500** together with the first and second sheaf-conveying rollers **114–117** and FIG. **24** is a schematic perspective view illustrating the construction of the stapler **500**.

The stapler **500** performs a stapling at certain positions of a sheaf nipped and conveyed by the first sheaf-conveying rollers **114**, **115** on the upstream side of the stapler **500** relative to the conveying direction. The stapler **500** comprises a head unit **501**, an anvil unit **502**, a supporting mechanism **520** which supports the units **501**, **502** such that the units **501**, **502** are freely moved in the orthogonal direction and rotated, a first drive mechanism **521** which moves the units **501**, **502**, and a second drive mechanism **522** which rotates the units **501**, **502**. In the stapler **500**, devices which engage or connect the head unit **501** with the anvil unit **502** do not transverse the sheet conveying path.

Further, the second sheet-conveying rollers **116**, **117** which convey the stapled sheaf and the second sensor **118** for fixing the stapling position of the sheaf (as will be specifically described herein below) are installed on the downstream side of the stapler **500**.

The head unit **501** separates one staple from a cartridge held within a cartridge case (not shown), bends the separated staple in the shape nearly resembling a letter U, and transfixes the sheaf with the bent staple. This unit **501** is provided with a sensor which detects the presence or absence of staple in the cartridge case.

The anvil unit **502** inwardly bends shanks of the staple which has penetrated through the sheaf and receives the shock of stapling performed by the head unit **501**. This unit **502** comprises a receiving plate, which inwardly bends the shanks of the staple, and a supporting plate, which receives the shock of the stapling action.

The supporting mechanism **520**, as illustrated schematically in FIG. **24**, comprises a frame **510** provided with a pair of lateral wall **509***a*, **509***b* and supporting shafts **503**, **506** extending along the orthogonal direction and supported by the frame **510**. The distance between the lateral wall **509***a*, **509***b* of the frame **510** is set to surpass at least the length of a sheet in the orthogonal direction, which is passable. The supporting shafts **503**, **506** are each formed of a round bar. The supporting shaft **503** is inserted through the head unit **501** and the supporting shaft **506** is inserted through the anvil unit **502**. The units **501**, **502** are freely moved in the orthogonal direction along the supporting shafts **503** and **506** and are freely rotated respectively about the supporting shafts **503** and **506**, respectively.

The first drive mechanism **521** comprises a spiral shaft **504** inserted through the head unit **501** and a spiral shaft **507** inserted through the anvil unit **502**. The spiral shafts **504**, **507** extend along the orthogonal direction and supported by the frame **510**. In consequence of the rotation of the spiral shaft **504**, the head unit **501** is moved in the orthogonal direction as guided by the supporting shaft **503**. In consequence of the rotation of the spiral shaft **507**, the anvil unit **502** is moved in the orthogonal direction as guided by the supporting shaft **506**.

The second drive mechanism **522** comprises a drive shaft **505** inserted through the head unit **501** and a drive shaft **508** inserted through the anvil unit **502**. The drive shafts **505**, **508** extend along the orthogonal direction and supported by the frame **510**. In consequence of the rotation of the drive shaft **505**, the driving force for transfixing a sheaf is transmitted to the head unit **501** and the head unit **501** is rotated about the supporting shaft **503** as a center. In consequence of the rotation of the drive shaft **508**, the driving force for bending shanks of a staple is transmitted to the anvil unit **502** and the anvil unit **502** is rotated about the supporting shaft **506** as a center. The drive shafts **505**, **508** include a shaft possessed of a rectangular cross section incapable of generating slippage for the purpose of infallibly transmitting the driving force to the units **501**, **502**. When the drive shafts are formed of a round bar, the slippage between the drive shafts and the units **501** and **502** may be precluded by means of a key or a key groove, for example.

The units **501**, **502** can be linearly moved independently and parallel along the orthogonal direction with the aid of the plurality of shafts **503–505** and **506–508**, which are inserted respectively.

The head unit **501** and the anvil unit **502** are moved along the orthogonal direction by the rotation of the spiral shafts **504**, **507** which have the same phases. A timing belt **511** is suspended as passed around the spiral shafts **504**, **507**. This belt **511** is connected to a drive motor **512**. The drive motor **512** is formed of a DC motor and enabled by a pulse disc sensor **513** to produce a controlled rotation. Owing to the construction, the units **501**, **502** can be severally moved in an equal distance. The first drive mechanism **521** is composed of the spiral shafts **504** and **507**, the timing belt **511**, the drive motor **521**, etc.

A light-permeable sensor **516** is mounted on the frame **510** for detecting the home positions of the units **501**, **502**. After detecting the gobos provided on the head unit **501** by the sensor **516**, the units **501**, **502** are both moved to the respective home positions. The distances of movement of the units **501**, **502** are set on the basis of the home positions.

The head unit **501** and the anvil unit **502** are actuated to produce the transfixing motion by the rotation of the drive shafts **505**, **508**. A belt **514** is suspended as passed around the drive shafts **505**, **508**. This belt **514** is connected to a drive motor **515**. Owing to this construction, the units **501**, **502** are severally driven to transfix a sheaf at positions arbitrarily

selected in the orthogonal direction. The second drive mechanism **522** is composed of the drive shafts **505** and **508**, the belt **514**, the drive motor **515**, etc.

[Description of operation]

The head unit **501** and the anvil unit **502** of the stapler **500** at first stand at rest at the home positions for intercepting the light from the sensor **516**. The sheets outputted from the copying machine **10** are conveyed to the additional-work tray **401** and are stacked and aligned. When as many sheets as suffice for one job are stacked on the additional-work tray **401**, the stacked sheet are conveyed as a sheaf in the direction of the stapler **500**.

The first sheaf-conveying rollers **114, 115** as a conveying device for nipping and conveying the sheaf to the stapler **500** can control the conveying distance of the sheaf by the amounts of their rotation. The first sheaf-conveying rollers **114, 115** convey the sheaf at a position such that the stapling position arbitrarily selected on the sheaf coincides with the transfixing position.

Thereafter, the drive motor **512** is actuated to rotate the spiral shafts **504, 507** through the belt **511** while the pulse disc sensor **513** detects the amount of rotation. The units **501, 502** are severally moved over an equal distance in the direction of the stapling positions selected arbitrarily. When the units **501, 502** are stopped at the selected stapling positions, the drive motor **515** is actuated to rotate the drive shafts **505, 508** through the belt **514**. The units **501, 502** are rotated to transfix a sheaf.

When the stapling is performed at a plurality of points falling on a straight line along the orthogonal direction, the units **501, 502** are moved to the next transfixing point by the operation of the motor **512** after completing the transfixing work at the first point. Then, the motor **515** is actuated to perform the transfixing work. By repeating this process, the stapling work at the plurality of points is wholly completed.

[Mechanism for conveyance of sheaf]

FIG. **25**A–FIG. **25**C are structural diagrams illustrating the first sheaf-conveying rollers **114, 115**.

The first sheaf-conveying rollers **114, 115** formed of a pair of upper and lower rollers are disposed in the upstream section of the stapler **500** and the second sheet-conveying rollers **116, 117** likewise formed of a pair of upper and lower rollers are disposed in the downstream section as illustrated in FIG. **23**. The first sheaf-conveying rollers **114, 115** nip and convey a sheaf awaiting a stapling and the second sheet-conveying rollers **116, 117** mainly nip and convey the stapled sheaf. The distance between the nipping position of the first sheaf-conveying rollers **114, 115** and the nipping position of the second sheet-conveying rollers **116, 117** is set at a size slightly smaller than the smallest of the sizes of sheets to be conveyed.

The upper roller **115** of the first sheaf-conveying rollers is freely pressed against and separated from the lower roller **114** of the first sheaf-conveying rollers by the operation of the first DC motor. The upper and lower rollers **114, 115** are both rotated by a stepping motor (denoted by reference numeral "**128**" in FIG. **31** which will be specifically described herein below). The conveying distance of the sheaf depends on the amount of rotation of the stepping motor. The second sheet-conveying rollers **116, 117** are similarly constructed. By the actuation of the second DC motor, the upper roller **117** is freely pressed against and separated from the lower roller **116**, independently of the first sheaf-conveying rollers **114, 115**. The upper and lower rollers **116, 117** are rotated by the same stepping motor as is used for driving the rollers **114, 115** and control the conveying distance of the sheaf.

The upper and lower rollers **114, 115** disposed on the upstream side of the stapler **500** are formed of a same kind of material having a desired hardness and are formed in an equal diameter. Likewise, the upper and lower rollers **116, 117** on the downstream side are formed of a same kind of material having a desired hardness and are formed in an equal diameter. However, the rollers **116, 117** have a smaller diameter than the rollers **114** and **115**.

To be more specific, the upper and lower rollers **114, 115** on the upstream side are constructed of solid rubber having hardness of not more than 18 Hs (JIS [Japanese Industrial Standard] A) and are amply deformed when the rollers **114, 115** are pressed against the sheaf. The pressing force by the upper and lower rollers **114, 115** is such that the width of contact between the rollers and the sheets exceeds **5**mm along the direction of rotation of the rollers. The upper and lower rollers **116, 117** on the downstream side are constructed of foam rubber of low hardness and exert on the sheaf lower pressing force than that on the upstream side. The hardness of the rollers **114, 115** will be further described herein below.

The lower roller **114** and the upper roller **115** of the first sheaf-conveying rollers are connected through a drive transmission mechanism **131***a* possessed of at least one idle gear **135** as illustrated in FIG. **25**A. The lower roller **114** is disposed such that the roller surface protrudes from the stacking base of the additional-work tray **401**. The drive transmission mechanism **131***a* is possessed of a link mechanism **560** which connects supporting shafts **135***a*, **114***a* and **115***a* respectively of the idle gear **135**, the lower roller **114**, and the upper roller **115**. The link mechanism **560** regulates the distance between the shafts of the idle gear **135** and the lower roller **114** and the distance between the shafts of the idle gear **135** and the upper roller **115**. An oblong hole **561** is formed in a (not shown) casing which supports the supporting shaft **114***a* of the lower roller **114** in a freely rotating state. The supporting shaft **115***a* of the upper roller **115** is slidably inserted through this oblong hole **561**. The oblong hole **561** extends in the direction perpendicular to the stacking base of the additional-work tray **401**. When the link mechanism **560** is actuated by the operation of the first DC motor, the upper roller **115** is moved along the direction perpendicular to the stacking base of the additional-work tray **401** between the spaced position (FIG. **25**B) and the pressing position (FIG. **25**C), with the supporting shaft **115***a* guided in the oblong hole **561**.

The supporting shaft **115***a* of the upper roller **115** is connected to one end of a spring **562** for pressing force. The length of the oblong hole **561** is such that the supporting shaft **115***a* avoids contacting to the edge of the oblong hole **561** while the upper roller **115** is pressed against the lower roller **114**. Thus, the desired pressing force by the spring **562** is exclusively given to the upper roller **115**. The pressing force is given in the direction perpendicular to the sheet face of stacked sheets or the sheaf.

A belt **136** is suspended as passed around pulleys **563, 564** mounted on the supporting shafts **135***a*, **114***a* respectively of the idle gear **135** and the lower roller **114**. The idle gear **135** is engaged with a gear **565** mounted on the supporting shaft **115***a* of the upper roller **115**. The rotational driving force of the stepping motor is transmitted to the lower roller **114**. However, the rotational driving force is transmitted to the upper roller **115** even when the upper and lower rollers **114, 115** are not pressed against each other, owing to the above construction. The advantages of the system, which transmits the rotational driving force to both the upper and lower rollers **114, 115**, will be described herein below.

On the supporting shaft **115***a* of the upper roller **115**, at least one one-way clutch **134** which permits rotation exclusively in the direction indicated by an arrow in the diagram is mounted, as illustrated in FIG. **25**B and FIG. **25**C. This one-way clutch **134** keeps the upper roller **115** from rotating when the link mechanism **560** is actuated and the upper roller **115** is lowered from the spaced position to the pressing position. The second sheet-conveying rollers **116, 117** are similarly constructed though omitted from illustration.

The upper rollers **115, 117** are so constructed as to produce no rotation while being pressed against each other. Thus, the possibility of the sheets in the sheaf suffering from such inconveniences as disruption of alignment and sustentation of folds and wrinkles can be precluded when the sheaf is conveyed from the first sheaf-conveying rollers **114, 115** for nipping and conveying the sheaf before stapling to the second sheet-conveying rollers **116, 117** disposed on the downstream side of the stapler **500**. The upper rollers **115, 117** and the lower rollers **114, 116** can be rotated through the drive transmission mechanism **131***a* even when the upper rollers **115, 117** and the lower rollers **114, 116** are in a separated state. Thus, the sheaf of an arbitrary number of sheets not exceeding the largest number allowed for conveyance can be conveyed without incurring such inconveniences as irregularity or deviation.

Further, the above arrangement of the shape, material, and disposition of the upper and lower rollers **114, 115** for conveying the sheaf makes it possible to convey the sheaf without entraining such inconveniences as disruption of alignment and sustentation of folds and wrinkles. Particularly, the setting of the material and the pressing force of the first sheaf-conveying rollers **114, 115** for conveying the sheaf awaiting a stapling makes it to convey the sheaf to the desired stapling position without incurring disruption of alignment. Further, the arrangement of the material and the pressing force of the-second sheet-conveying rollers **116, 117** for mainly conveying the stapled sheaf makes it possible to convey the sheaf without incurring such inconveniences as misalignment and wrinkles even when the sheaf thrust into the interface between the rollers **116** and **117** which are in a mutually pressed state. The construction of the drive mechanism for the second sheet-conveying rollers **116, 117** is identical to that of the drive mechanism for the first sheaf-conveying rollers **114, 115**. Thus, there is no possibility of the sheaf being rotated about a staple as a center and no possibility of the sheets in the sheaf incurring such inconveniences as misalignment and wrinkles around the staple when the sheaf has been stapled only at one point.

A first sensor **137**, which detects the edge of the sheaf being conveyed, is disposed near the downstream side of the first sheaf-conveying rollers **114, 115** as illustrated in FIG. **23**. Likewise, the second sensor **118** is disposed near the downstream side of the second sheet-conveying rollers **116, 117**. The sensors **118, 137** are each disposed at a position separated by a certain distance from the position for driving a staple needle.

At least the conveying path between the first sheaf-conveying rollers **114, 115** and the second sensor **118** is formed of a guide in a straight shape. The reason for the use of the straight guide is as follows.

The leading end of the sheaf is aligned, during the temporary stacking of sheets, by the leading end stopper **409**. The pressure contact of the first sheaf-conveying rollers **114, 115** is initiated while the sheaf is in the state. Thus, the leading end of the sheaf is nipped as kept in the aligned state by the first sheaf-conveying rollers **114, 115**. The conveying path from the first sheaf-conveying rollers **114, 115** to the stapling position has a straight shape without bending. The leading end of the sheaf, therefore, keeps the aligned state intact even when the sheaf is nipped and conveyed to the stapling position by the first sheaf-conveying rollers **114, 115**. If the conveying path on the downstream side of the first sheaf-conveying rollers **114, 115** in the conveying direction is bent in the shape of an arc, the sheaf is elongated along the guide plate of a small radius and shortened along the guide plate of a large radius and the leading end of the sheaf is slanted relative to the guide plate. If the stapling perpendicular to the guide place is done, the sheaf is inevitably bound obliquely. Namely, the conveying path from the first sheaf-conveying rollers **114, 115** to the stapling position must be in a straight shape when the stapler **500** staples the sheaf which is nipped by the first sheaf-conveying rollers **114, 115**.

The present embodiment, as described specifically herein below, is constructed such that a sheaf is nipped and conveyed by the first sheaf-conveying rollers **114, 115** and the sheaf is further nipped and conveyed by the second sheet-conveying rollers **116, 117** and the sheaf is separated from the first sheaf-conveying rollers **114, 115** and the sheaf is continuously conveyed by the second sheet-conveying rollers **116, 117** only and then the sheaf is stapled by the stapler **500**. In other word, the leading end of the sheaf must remain in the aligned state until the sheaf being nipped and conveyed by the first sheaf-conveying rollers **114, 115** is newly nipped by the second sheet-conveying rollers **116, 117**. It is, thus, required that the conveying path from the first sheaf-conveying rollers **114, 115** to the second sensor **118** at which the second sheet-conveying rollers **116, 117** starts nipping the sheaf is in a straight shape.

The second sheet-conveying rollers **116, 117** nip the sheaf in the downstream side from the stapling position. Thus, the conveying path in the downstream side from the second sensor **118** does not need to be in a straight shape but may be bent in the shape of an arc, for example. The finisher as a whole, therefore, can be prevented from adding to the size.

[Advantages of system for giving rotational driving force to both upper and lower rollers conveying the sheaf and hardness of rollers]

The advantages of transmitting the rotational driving force to both the upper and lower rollers engaging in the conveyance of the sheaf will be described below. In this specification, for the sake of convenience of description, the form of giving rotational driving force to both upper and lower rollers will be defined as "forced-parallel movement".

The sheet deviation was measured based on the presence or absence of the forced-parallel movement. The sheet deviation Δd represents the difference (mm) between the leading end of the foremost sheet and the leading end of the hindmost sheet being conveyed along the conveying direction as illustrated in FIG. **26**. The measuring conditions were as follows.

1. Hardness of roller: 15 Hs (JIS A) as upper and lower rollers
2. Pressing force: 2 Kg
3. Method of conveyance: Manual feeding
4. Roller diameter: 30 mm
5. Conveying distance: 38 mm

The hardness of the rubber used for the rollers was measured by the spring type hardness test (Type A) specified in JIS K 6301.

The results of the test are shown in FIG. **27**A. It is clearly noted from this graph that the sheet deviation Δd in the absence of a forced-parallel movement was about 1.4 times

that in the presence of a forced-parallel movement. The data clearly show that the forced-parallel movement system of driving both the paired rollers allows more reduction in the sheet deviation than the system of driving one of the paired rollers and following the other roller.

Next, the hardness of the rollers for conveying the sheaf will be studied below.

The rollers with varying hardness were tested for sheet deviation Δd. The conditions of the test were as follows.

1. Identical upper and lower rollers and forced-parallel movement
2. Pressing force: 2 Kg
3. Speed of conveyance: 320 mm/sec
4. Roller diameter: 24 mm
5. Conveying distance: 38 mm

The sheet deviation Δd must be repressed to below 1 mm for obtaining a sheaf with a fine appearance after the stapling. Thus, the sheet deviation Δd within 1 mm were rated as acceptable. The results of the test are shown in FIG. **27**B.

It is clear from this graph that, in case of the silicone rubber rollers having 2 Hs (JIS A), 14 Hs (JIS A), and 18 Hs (JIS A) in hardness, the sheet deviation Δd was invariably less than 1 mm and were rated as acceptable. In case of the EPDM (ethylene propylene rubber) roller and the POM (polyacetal) rollers having both 60 Hs (JIS A) in hardness, the sheet deviation Δd were both more than 1.4 mm and were rated as not acceptable. In case of the silicone rubber roller having 27 Hs (JIS A) in hardness, the sheet deviation Δd at times exceeded 1 mm. The data clearly show that it suffices to use a roller, not more than 18 Hs (JIS A) in hardness, for the purpose of repressing the sheet deviation Δd to below 1 mm, with due allowance for more or less dispersion of test results.

[Control of stapling position]

When the staple mode is selected, sheets are stacked on the additional-work tray **401**. At this time, the first sheaf-conveying rollers **114, 115** are separated from each other. After the temporary stacking or storing of the sheets is completed, the first sheaf-conveying rollers **114, 115** are shifted to a mutually pressed state to nip a sheaf of the sheets and the leading end stopper **409** retracts outside the conveying path. Then, the sheaf is conveyed by rotating the first sheaf-conveying rollers **114, 115** and the stapling position is located along the conveying direction. The present embodiment contemplates three staple modes. The first mode is "leading end bind" which binds the leading end of the sheaf along the conveying direction. The second mode is "center bind" which binds the central section of the sheaf along the conveying direction. The third mode is "trailing end bind" which binds the trailing end of the sheaf along the conveying direction. The operation of the positioning depends on these modes. The each operation of the location for modes will be described below.

(1) Leading End Bind

FIG. **28**A–FIG. **28**F are explanatory diagrams illustrating the operation of leading end bind.

The leading end of the sheaf has already undergone the FD-alignment during the temporary stacking of sheets with the blocking plate **409***b* of the leading end stopper **409** used as a regulating face (FIG. **28**A). In the mode of leading end bind, it suffices for the location of the stapling position to convey the sheaf in a certain distance without reference to the size of sheet. To be specific, it is only required that the first sheaf-conveying rollers **114, 115** convey the sheaf in the distance resulting from adding the length from the leading end of the sheaf to the desired stapling position (normally about 10 mm) to the length from the blocking plate **409***b* of the leading end stopper **409** to the stapler **500** (FIG. **28**B). After the sheaf has been conveyed in the prescribed distance, the rollers **114, 115** are stopped and the stapler **500** is actuated to staple the sheaf (FIG. **28**C).

The conveyance of the sheaf is resumed after the completion of the stapling. The conveyance of the sheets is stopped when the leading end completely reaches the second sheet-conveying rollers **116, 117**. At this time, the second sheet-conveying rollers **116, 117** are still in a mutually separated state (FIG. **28**D). After the conveyance of the sheaf has ceased, the second sheet-conveying rollers **116, 117** are shifted to a mutually pressed state to nip the leading end of the sheaf. Then, the second sheet-conveying rollers **116, 117** are rotated to start the conveyance of the sheaf again (FIG. **28**E). The first DC motor is actuated with continuing the conveyance of the sheaf and exclusively shifts the first sheaf-conveying rollers **114, 115** to a mutually separated state (FIG. **28**F). The sheaf is subsequently conveyed and nipped by the second sheet-conveying rollers **116, 117** toward the accumulating tray unit **600**.

The stepping motor rotates the first and second sheaf-conveying rollers **114–117**. The conveying distance of the sheaf is controlled by regulating the pulses of the stepping motor.

(2) Center Bind

FIG. **29**A–FIG. **29**D are explanatory diagrams illustrating the operation of the center bind.

In the mode of center bind, the stapling is done in the central section of the sheaf along the conveying direction. Naturally, the conveying distance of the sheaf for the stapling varies with the size of sheet. The conveying distance is long as compared with that involved in the mode of leading end bind.

The stepping motor conveys the sheaf. It is theoretically possible to control, by simply changing pulses, the conveying distance even when the conveying distance is long. However, the diameters of the sheaf-conveying rollers **114–117** and the widths of the nips cannot be thoroughly freed from dimensional dispersions. Namely, the inaccuracy in the actual conveying distance enlarges in proportion as the conveying distance lengthens. To reduce the inaccuracy, the conveyance of the sheaf in the mode of center bind is effected as follows.

First, a sheaf is nipped and conveyed by the first sheaf-conveying rollers **114, 115**. After the second sensor **118** disposed in the downstream side of the second sheet-conveying roller **116, 117** has detected the leading end of the sheaf, the sheaf is further conveyed in a distance proper for the sheet size and is stopped (FIG. **29**A and FIG. **29**B). Then, the sheaf is stapled (FIG. **29**C).

At this time, the leading end of the sheaf has completely reached the second sheet-conveying rollers **116, 117**. The second sheet-conveying rollers **116, 117** nips the sheaf. Then, the second sheet-conveying rollers **116, 117** are rotated to resume the conveyance of the sheaf. Meanwhile the first DC motor is actuated to shift the first sheaf-conveying rollers **114, 115** alone to a mutually separated state, continuing the conveyance of the sheaf (FIG. **29**D). Thereafter, the sheaf is conveyed and nipped by the second sheet-conveying rollers **116, 117** toward the accumulating tray unit **600**.

The center bind does not need to be limited to the above manner but may be carried out as follows instead. First, the sheaf is nipped and conveyed by the first sheaf-conveying rollers **114, 115**. The conveyance is stopped when the

leading end of the sheaf completely reaches the second sheet-conveying rollers **116**, **117**. And the second sheet-conveying rollers **116**, **117** nips the sheaf. After the completion of the nipping by the second sheet-conveying rollers **116**, **117**, the first DC motor is actuated to shift the first sheaf-conveying rollers **114**, **115** to a mutually separated state.

After the completion of the operation of mutually separating the first sheaf-conveying rollers **114**, **115**, the second sheet-conveying rollers **116**, **117** is rotated to resume the conveyance of the sheaf. When the second sensor **118** detects the leading end of the sheaf, the sheaf is stopped after conveyed in a proper distance in response to the sheet size. Then, the sheaf is stapled. The stapled sheaf resumes being conveyed and nipped by the second sheet-conveying rollers **116**, **117** toward the accumulating tray unit **600**. In short, the sheaf for the binding work can be conveyed as pulled by the second sheet-conveying rollers **116**, **117** and, thus, the leading end of the sheaf does not form resistance and the irregularity of the leading end of the sheaf is reduced.

The mode of center bind is applied solely to creased sheets. And sheets having a length not less than twice the length of a sheet of the smallest size to be conveyed are only applicable.

(3) Trailing End Bind

FIG. **30**A–FIG. **30**D are explanatory diagrams illustrating the operation of trailing end bind.

In the mode of trailing end bind, first the sheaf is nipped and conveyed by the first sheaf-conveying rollers **114**, **115**. When the leading end of the sheaf completely reaches the second sheet-conveying rollers **116**, **117**, the conveyance is stopped and the sheaf is nipped by the second sheet-conveying rollers **116**, **117** (FIG. **30**A). After the completion of the nipping by the second sheet-conveying rollers **116**, **117**, the first DC motor is actuated to shift the first sheaf-conveying rollers **114**, **115** to a mutually separated state (FIG. **30**B). After the completion of the operation for mutually separating the first sheaf-conveying rollers **114**, **115**, the second sheet-conveying rollers **116**, **117** is rotated to resume the conveyance of the sheaf (FIG. **30**C). When the second sensor **118** detects the leading end of the sheaf, the sheaf is stopped after conveyed in a certain distance proper for the sheet size. Then the sheaf is stapled (FIG. **30**D).

The stapled sheaf resumes being conveyed and nipped by the second sheet-conveying rollers **116**, **117** toward the accumulating tray unit **600**.

In the above mode of conveyance, the conveying distance is set based on the position of the second sensor **118**. Optionally, the conveying distance in the mode of trailing end bind may be set based on the position of the first sensor **137** which is disposed in the downstream side of the first sheaf-conveying rollers **114**, **115**. In the present mode, the sheaf is conveyed in a certain distance after the first sensor **137** has detected the trailing end of the sheaf. Namely, the sheaf has only to be conveyed in a prescribed distance without reference to the size of sheet. The first sensor **137** approximates closely to the stapling position. Advantageously, it results in shortening the conveying distance and improving the positioning accuracy.

Incidentally, the following operational flow is conceivable for the purpose of shortening the total time to be spent in conveying the sheaf and improving the productivity. The operational flow specifically comprises a step of causing the first sheaf-conveying rollers **114**, **115** located in the upstream side and the second sheet-conveying rollers **116**, **117** located in the downstream side of the stapler **500** severally to nip and convey the sheaf again, and a step of mutually separating the first sheaf-conveying rollers **114**, **115** during the resumed conveyance.

When a sheaf has already stapled as in the mode of leading end bind or the mode of center bind, it incurs no particularly serious problem that the sheaf resumes being conveyed and nipped by both of the sheaf-conveying rollers **114**, **115** and **116**, **117** and then the first sheaf-conveying rollers **114**, **115** is mutually separated. However, it possibly incur such inconveniences as disruption of sheet alignment owing to the difference in the conveying speed of the sheaf in the upstream zone and the downstream zone that the sheaf which has not been stapled is nipped and conveyed by both the sheaf-conveying rollers **114**, **115** and **116**, **117** as in the mode of trailing end bind.

Accordingly, in the present embodiment the first DC motor is actuated after the completion of nipping the sheaf by the second sheet-conveying rollers **116**, **117** and the sheaf resumes being conveyed by the second sheet-conveying rollers **116**, **117** alone after the completion of the mutual separation of the first sheaf-conveying rollers **114**, **115** when the sheaf will be stapled later.

[Retracting of stapler **500** during restoring from jam]

The head unit **501** and the anvil unit **502** of the stapler **500** are so constructed as to be moved in the orthogonal direction by the drive motor **512** as a DC motor. The drive motor **512** is provided with the pulse disc sensor **513** as a pulse generating device and controls the positions of the units **501**, **502** based on the pulses outputted from the pulse disc sensor **513** and the signal outputted from the sensor **516** which detects the units **501**, **502** at the home positions. The home positions of the units **501**, **502** are the positions approximated most closely to the front face of the finisher **100** inside the frame **510**, i.e. the positions at which the units **501**, **502** are kept waiting outside the conveying path.

When a jam of sheet occurs in the stapler **500** while the stapling is being performed on the sheaf which have been stacked in the additional-work tray **401** and conveyed to the stapler **500**, the units **501**, **502** respectively are returned to the home positions and then the fact that the jam of sheet has occurred in the stapler is outputted on the control panel of the copying machine **10**.

When the completion of restoration from the jam is detected, the units **501**, **502**, which have retracted to the home positions, are moved to the positions which they assumed when the jam of sheet occurred.

<<Sheet discharge unit **550**>>

FIG. **31** is a perspective view illustrating an artist concept of a sheet discharge unit **550** which conveys a stapled sheaf of sheets and an unstapled single sheet toward the accumulating tray unit **600**. In the diagram, the reference numeral "**132**" de notes a conveying path for conveying the single sheet and the reference numeral "**133**" denotes a conveying path for the sheaf. In this diagram, the positional relation of the rollers is deliberately differentiated from that illustrated in FIG. **2** to facilitate comprehension of the conveying paths.

The accumulating tray **601** of the accumulating tray unit **600** receives a sheaf of sheets, which is discharged from the additional-work tray **401** and stapled by the stapler **500**, and an unstapled single sheet, which is conveyed through the other conveying path. The sheet discharge unit **550** is provided for the purpose of conveying the sheaf and the single sheet.

The sheet discharge unit **550**, as illustrated in the diagram, comprises the third sheet-conveying rollers **119**, **120** which conveys the sheaf, the conveying roller **121** disposed in the downstream side of the switch claw **103** and conveys a lone sheet, and discharging rollers **122**, **123** which outputs the

sheaf or the single sheet into the accumulating tray **601** in addition to the first and second sheaf-conveying rollers **114**, **115** and **116**, **117**.

A DC motor **130** independently of the other rollers actuates the discharging rollers **122**, **123**. The DC motor **130** includes a disc **551**. The rotational speeds of the discharging rollers **122**, **123** are controlled in accordance with pulses outputted from the disc **551** detected by a pulse disc sensor **552**.

The first and second and third sheaf-conveying rollers **114**, **115** and **116**, **117** and **119**, **120** are driven by one stepping motor **128** through a belt **553**. The third sheet-conveying rollers **119**, **120** are connected to the stepping motor **128** through a one-way clutch **129** provided on the shaft of the roller **120**. The one-way clutch **129** rotates freely in the direction of permitting the sheaf to move along the conveying direction even when the stepping motor **128** is in a stopped state. The other rollers disposed in the sheet-conveying path such as the conveying roller **121** are altogether driven by another DC motor (not shown).

The discharging rollers **122**, **123** are required to steadily convey a lone unstapled sheet or a stapled sheaf, which are different in thickness. Accordingly, the discharging rollers **122**, **123** comprise rollers made of a material of low hardness, and a clearance of the upper rollers **123** is large enough for accepting a thick sheaf, and pressing force to the lower roller **122** is relatively weak. The sheet discharge unit **550** is provided with drive transmission mechanisms **131***a*–**131***d* including at least one idle device capable of transmitting the driving motions of the lower rollers **114**, **116**, **120** and **122** to the upper rollers **115**, **117**, **119** and **123** respectively in order to convey both the upper and lower sections of the sheaf steadily.

[Discharge of sheaf or single sheet onto accumulating tray **601**]

The sheaf stored on the additional-work tray **401** is nipped and conveyed to the stapling position by the first sheaf-conveying rollers **114**, **115** or the second sheet-conveying rollers **116**, **117**, depending on the selected mode of staple. After the stapling, the conveyance is started again by the second sheet-conveying rollers **116**, **117**. One stepping motor **128** rotates the first and second sheaf-conveying rollers **114**, **115** and **116**, **117**. This stepping motor **128** also rotates the third sheet-conveying rollers **119**, **120**. The sheaf-conveying path **133** joins the sheet-conveying path **132** in the downstream side of the third sheet-conveying rollers **119**, **120**. Accordingly, the sheaf passes through the discharging rollers **122**, **123** and reaches the accumulating tray **601**. The discharging rollers **122**, **123** are rotated independently by the DC motor **130**. The rotational speeds of the discharging rollers **122**, **123** are controlled, depending on the pulses outputted from the disc **551**.

The stapled sheaf is conveyed through the conveying path **133**. After the leading end of the sheaf has been completely nipped by the third sheet-conveying rollers **119**, **120** with the one-way clutch **129**, the second sheet-conveying rollers **116**, **117** are mutually separated. The first sheaf-conveying rollers **114**, **115** have been already separated mutually by the time that the leading end of the sheaf has been completely nipped in the third sheet-conveying rollers **119**, **120**.

When the first sensor **137** detects the fact that the trailing end of the sheaf has passed the leading end stopper **409**, the leading end stopper **409** is reset to close the sheet discharging outlet **401***a* of the additional-work tray **401**. Then, the temporary accumulation of sheets for the next stapling, or the next job is started.

The stepping motor **128** is stopped after the sheaf has been further conveyed and the leading end of the sheaf has been completely nipped in the discharging rollers **122**, **123**. At this time, the rotation of the discharging rollers **122**, **123** has been already started and the first and second sheaf-conveying rollers **114**–**117** are in a mutually separated state while the third sheet-conveying rollers **119**, **120** are provided with the one-way clutch **129**. Thus, the sheaf is continuously conveyed and stored in the accumulating tray **601**.

The distances between the leading end stopper **409** and the discharging rollers **122**, **123** are set such that the leading end of the sheaf in the preceding job can completely reach the discharging rollers **122**, **123** before the completion of the temporary accumulation of the sheets of the next job, without reference to the size of sheet and the number of sheets. Therefore, the stepping motor **128** is standing at rest at the time that the temporary accumulation of the sheets for the next job is completed. In brief, the first sheaf-conveying rollers **114**, **115** can be pressed against the sheaf at the time that the accumulation of the sheets for the next job is completed and the start of stapling the next sheaf does not need to be retarded.

<<Accumulating tray unit **600**>>

[Whole construction of accumulating tray unit **600**]

FIG. **32** is a structural diagram illustrating the accumulating tray unit **600** and FIG. **33** is a partially cutaway bottom view illustrating the accumulating tray **601** of the accumulating tray unit **600**. A sheaf of sheets or lone sheet is successively outputted into the accumulating tray unit **600**. The sheaf or the lone sheet will be expressed hereinafter as "sheet/sheaf" for the sake of the convenience of description.

The accumulating tray unit **600**, as illustrated in FIG. **32**, comprises the accumulating tray **601** which stores the sheet/sheaf and moves up and down proportionate to the amount of accumulation, an elevating mechanism which raises and lowers the accumulating tray **601**, an angle adjusting device **602** which adjusts the angle of the tray (the angle of inclination of the accumulating base relative to the horizontal position), depending on the condition of the additional-working performed on the discharged sheet, an empty sensor **605** which detects the presence or absence of the sheet/sheaf on the accumulating tray **601**, and an upper face sensor **606** which detects the upper face of the sheet/sheaf stacked on the tray **601**. The discharged sheet sensor **124** is disposed on the upstream side of the discharging rollers **122**, **123**.

The sheet/sheaf given various additional-workings (folding, punching and stapling) and the sheet/sheaf without no additional-workings after outputted from the copying machine **10** are discharged onto the accumulating tray **601**. The accumulating tray **601** is movable up and down and can store a large number of sheet/sheaf. The accumulating tray **601** is formed in a shape such that the leading end (the left end in FIG. **32**) is raised, and can secure a perfect property of either discharging or stacking such sheets with no fold. The accumulating tray **601**, as illustrated in FIG. **33**, has a larger width than the acceptable largest width of sheet and has the opposite ends, in the width direction of the basal section, retained with a retainer (not shown).

The elevating mechanism includes a reversible motor (not shown) which raises and lowers the accumulating tray **601**, a guide rail, etc. This construction is well known in the art, it will be omitted from the description here.

The empty sensor **605** and the upper face sensor **606** are each formed of a transmission type photosensor provided with a light-emitting device and a light-sensitive device. The light-emitting device and the light-sensitive device of the empty sensor **605**, as illustrated in FIG. **32**, are vertically disposed as opposed to each other across the accumulating

tray **601**, and possessed of optical axes which intersect the stacking base via a through hole **610** (FIG. **33**) formed in the accumulating tray **601**.

The light-emitting device and the light-sensitive device of the upper face sensor **606**, as illustrated likewise in FIG. **33**, are disposed on the base of the accumulating tray **601** so as to intersect the upper section of the accumulating tray **601** in the width direction, and are possessed of optical axes extending along the width direction or CD-direction. The upper face sensor **606** is mounted on a supporting plate **634**, which is raised from the casing of the finisher **100**. The upper face sensor **606** is not vertically movable. The accumulating tray **601** is moved up and down by the elevating mechanism while the upper face sensor **606** detects the upper face of the sheet on the accumulating tray **601**. Namely, the drop distance of the sheet/sheaf from the nip part of the discharging rollers **122**, **123** is kept constant without reference to the amount of sheets stacked on the accumulating tray **601**.

The angle adjusting device **602**, as illustrated in FIG. **32**, comprises a movable plate **620** which is mounted rotatably to the accumulating tray **601** and protrudes freely from the stacking base, a cam **603** which contacts to the lower face of the movable plate **620** and rotates in one direction, and a drive motor **604** which rotates the cam **603**. The amount of the protrusion of the movable plate **620** varies in proportion to the amount of rotation of the cam **603**. The inclination of the tray is adjusted to a desired angle based on the above mechanism. The upper face of the movable plate **620** and the direction in which the sheet is discharged by the discharging rollers **122**, **123** are nearly parallel when the movable plate **620** is elevated to the upper limit. The elevation of the accumulating tray **601** is controlled by actuating the elevating mechanism based on the signals from the discharged sheet sensor **124**, the upper face sensor **606**, and the empty sensor **605**.

[Operation of accumulating tray unit **600**]

FIG. **34**A and FIG. **34**B are respectively a flow chart illustrating a control routine for detecting the upper face of sheet/sheaf and a control routine for moving the accumulating tray **601** down with a drive motor in a series of operations of the accumulating tray unit **600**.

The operation of the accumulating tray unit **600** will be described below with respect to the case (1) of discharging unfolded sheets one by one and the case (2) of discharging a sheaf obtained by subjecting sheets without a crease to either leading end fold or trailing end fold. The operation involved in the case of discharging a sheaf centrally creased and bounded like a weekly magazine will be described herein below.

(1) Case of Discharging Unfolded Sheets One by One

The accumulating tray **601** is elevated by the elevating mechanism when the empty sensor **605** detects the absence of a sheet on the accumulating tray **601**. The elevating mechanism is stopped as soon as the upper face of the accumulating tray **601** intercepts the light incident on the upper face sensor **606**. As a result, the accumulating tray **601** is kept at a lower position separated by a certain distance from the nip part of the discharging rollers **122**, **123**. And the accumulating tray **601** is kept waiting at the position, or the initial position until the sheet is discharged.

When the sheet is discharged onto the accumulating tray **601**, the empty sensor **605** judges that the sheet exists. The accumulating tray **601** is gradually lowered by the elevating mechanism under the condition that the sheet exists on the accumulating tray **601**. The elevating mechanism is stopped as soon as the interception of the light incident on the upper face sensor **606** is released.

To be more specific, the timer is started when the upper face sensor **606** detects the sheet as illustrated in FIG. **34**A, namely when the stacked sheets intercept the light incident on the upper face sensor **606** and the sensor **606** becomes being in ON-state (**S51** and **S52**). The upper face detection flag is set to be "1" when the upper face sensor **606** continuously detects the sheet for the duration of T1 [second] during a certain period following the detection of the trailing end of the sheet by the discharged sheet sensor **124**, wherein **T1** is shorter than the certain period ("Y" at **S53**, and **S54**). The timer is reset when the upper face sensor **606** does not continuously detect the sheet for the duration of T1 [second] (**S55**). When the upper face detection flag is "1" ("Y" at **S61**) as illustrated in FIG. **34**B, it is judged whether or not the upper face sensor **606** has detected the sheet (**S62**). When the judgment is affirmative, the drive motor is actuated to lower the accumulating tray **601** ("Y" at **S62**, **S63**). When the interception of the light incident on the upper face sensor **606** is released and the upper face sensor **606** becomes being in OFF-state in consequence of the downward movement of the accumulating tray **601**, the drive motor is stopped and the upper face detection flag is reset ("N" at **S62**, **S64**, **S65**)

When the discharge of sheets onto the accumulating tray **601** continues, the upper face sensor **606** is again hidden from the incident light by the stacked sheets. Then, the accumulating tray **601** is again lowered until the interception of the light incident on the upper face sensor **606** is released.

By repeating the steps, the distance between the nip part of the discharging rollers **122**, **123** which have fixed positions and the uppermost face of the stacked sheets is kept to equal the distance at the initial position even when the number of stacked sheets happens to be large. Thus, the sheets can be always stacked steadily on the accumulating tray **602** without impairing the property of discharging sheets even when the number of sheets stacked on the accumulating tray **601** is large.

When the sheets on the accumulating tray **601** are removed, the empty sensor **605** detects the absence of a sheet and the accumulating tray **601** is elevated by reversing the rotation of the drive motor. The elevation of the accumulating tray **601** is stopped by halting the rotation of the drive motor when the upper face sensor **606** detects the upper face of the accumulating tray **601**. Namely, the accumulating tray **601** is returned to be at the initial position for retaining desired distances between the tray **601** and the discharging rollers **122**, **123**.

(2) Case of Discharging Sheaf Obtained by Subjecting Sheets Without a Crease to Leading End Bind or Trailing End Bind

In this case, the movable plate **620** of the angle adjusting device **602** is moved by the driving motor **604** until the face for receiving sheets is nearly leveled as illustrated in FIG. **32**. After the movable plate **620** has been moved, the accumulating tray **601** is vertically moved until the upper face of the movable plate **620** reaches the position of the upper face sensor **606**. As a result, the movable plate **620** assumes a position such that the sheet receiving face nearly parallel aligns with the direction of sheets discharged by the discharging rollers **122**, **123**. The operation of moving the movable plate **620** and the accumulating tray **601** is completed at least before the leading end of the first stapled sheaf reaches the discharging rollers **122**, **123**.

Thereafter, the sheaf is discharged onto the sheet receiving face of the movable plate **620** as kept nearly parallel to the discharging direction by the discharging rollers **122**, **123**. When the discharged sheaf intercepts the light incident on

the upper face sensor **606**, the accumulating tray **601** is lowered to a position such that the intersection of light incident on the upper face sensor **606** is released. It results in making it possible to discharge the stapled sheaf for the next jog under substantially the same condition as used for the sheaf for the previous job.

The sheaf is discharged, substantially parallel to the sheaf which has been already stacked on the accumulating tray **601**. It results in preventing the leading end and the corners of the sheaf being discharged from contacting to a staple of the sheaf or reducing a shock in contacting to the staple. For that reason, such inconveniences as discharge failure of a sheaf of sheets, damage of the sheet as folds in corners, or misalignment of the sheet or the sheaf are no longer occurred.

Further, the discharge of unfolded sheets is attained without moving the movable plate **620** upward. Accordingly, both the discharge of unstapled sheets and the discharge of a stapled sheaf can be carried out satisfactorily on the single accumulating tray **601**.

The accumulating tray **601** is moved upward and returned to the home position when the sheaf on the accumulating tray **601** is removed.

<<Guiding unit **160**>>

[Construction of guide unit **160**]

FIG. **35**A is a schematic structural diagram illustrating an auxiliary guide of the guide unit, FIG. **35**B is an explanatory diagram illustrating discharge failure of a sheaf centrally creased and bound like a weekly magazine, and FIG. **36** is a perspective view illustrating the auxiliary guide.

The additional-worked sheaves include a so-called "weekly-magazine-like sheaf" which results from centrally creasing a sheet, stacking the sheet and centrally stapling the resultant sheaf. The weekly-magazine-like sheaf is stored on the accumulating tray **601** in an opened state that the stapled creases form a ridge **633**, as illustrated in FIG. **35**B. The discharging rollers **122**, **123** discharge the weekly-magazine-like sheaf onto the accumulating tray **601** while the creases of the weekly-magazine-like sheaf rise. As soon as the creased central section of this sheaf passes through the nip part of the discharging rollers **122**, **123**, the leading end of the sheaf hangs down. A weekly magazine-like sheaf as have been stacked on the accumulating tray **601** has the possibility that the suspended leading end of the subsequently discharged weekly-magazine-like sheaf contacts and engages with the vicinities of the ridge **633** or the central raised section of the stacked weekly-magazine-like sheaf and thereby discharge failure of the subsequent sheaf is caused. To preclude the occurrence of such inconveniences, it is necessary that the leading end of the weekly-magazine-like sheaf being discharged should fall on the further downstream side of the ridge **633** of the weekly-magazine-like sheaf stacked on the accumulating tray **601** along the discharging direction.

From this point of view, the finisher **100** of the present embodiment is provided with the guide unit **160** which supports the lower face of the weekly-magazine-like sheaf freshly discharged from the discharging rollers **122**, **123** as illustrated in FIG. **35**A and FIG. **36**. This guide unit **160** comprises an auxiliary guide **125** which is movable in a horizontal direction toward or away from the downstream side of the discharging rollers **122**, **123** and a driving mechanism which moves the auxiliary guide **125** forward or backward.

The auxiliary guide **125** is constructed of a plate shaped nearly like a comb so as to avoid interfering with the lower discharging roller **122**. The auxiliary guide **125** is disposed beneath the discharging rollers **122**, **123**. The auxiliary guide **125** moves forward or backward in a horizontal direction between the hindmost position at which the leading end is located on the upstream side from the nip position of the discharging rollers **122**, **123** and the foremost position at which the leading end is located on the downstream side from the nip position. The foremost position of the auxiliary guide **125** is set such that the leading end of the weekly-magazine-like sheaf is discharged to pass over the ridge **633** of the weekly-magazine-like sheaf on the accumulating tray **601**.

A rack (not shown) is integrally mounted to the auxiliary guide **125**. The auxiliary guide **125** is moved forward or backward by transmitting the rotation of a motor **127** to the auxiliary guide **125** through the rack. The drive mechanism is composed of the rack, the motor **127**, etc.

The auxiliary guide **125** is driven by the motor **127** to move the foremost position when the weekly-magazine-like sheaf is discharged. The weekly-magazine-like sheaf is discharged from the discharging rollers **122**, **123**, with the lower face supported by the auxiliary guide **125**. The leading end of the sheaf falls on the further downstream side along the discharging direction from the ridge **633** of the sheaf on the accumulating tray **601**.

[Operation of auxiliary guide **125**]

FIG. **37** is a flowchart showing the steps of the operation of the guide unit **160**.

The guide unit **160** is used exclusively when the mode of center bind staple or weekly-magazine-like sheaf is selected. When it is detected that the mode of center bind staple has been selected ("Y" at S**71**), the sheaf is aligned on the additional-work tray **401** and it is judged whether or not the stapler **500** has completed the stapling (S**72**). At the time that the stapling is completed ("Y" at S**72**), the motor **127** is actuated to move the auxiliary guide **125** to the foremost position so as to partly cover the upper face of the accumulating tray **601** (S**73**), as illustrated in FIG. **35**A and FIG. **36**. Besides, the sheaf has already been conveyed by the third sheet-conveying rollers **119**, **120** after the completion of stapling.

The timer is started when the discharged sheet sensor **124** detects the leading end of the sheaf ("Y" at S**74**, S**75**). The protrusion of the auxiliary guide **125** is retained until the timer counts up a certain time T2 (S**75**, "N" at S**76**). The time T2 is sufficient for the leading end of the sheaf being discharged to pass over the ridge **633** of the sheaf already stored in the accumulating tray **601**.

The weekly-magazine-like sheaf is discharged as nipped by the discharging rollers **122**, **123**. The auxiliary guide **125** supports the lower face of the weekly-magazine-like sheaf being discharged. There is no possibility of the leading end hanging down. As a result, the weekly-magazine-like sheaf being discharged advances on the auxiliary guide **125** and cannot contact to the sheaf already stacked on the accumulating tray **601**. The leading end of the sheaf being discharged infallibly falls on the further downstream side along the discharging direction from the ridge **633** of the sheaf on the accumulating tray **601**. Namely, the leading end of the sheaf being discharged avoids contacting to the ridge **633** of the stacked sheaf and the defective discharge of the sheaf is precluded.

When the timer counts up the time T2 ("Y" at S**76**), the auxiliary guide **125** retracts to the home position (S**77**) and the sheaf being discharged falls in an unconstrained state onto the accumulating tray **601**. The weekly-magazine-like sheaf for the next job is received and stored by the same steps.

In the above manner, the finisher **100** can secure perfectly the property of smoothly discharging a weekly-magazine-like sheaf. The auxiliary guide **125** can retract to the home position incapable of interfering with the discharged sheets. The accumulating tray **601** is allowed to keep the shape intact and to secure perfectly the property of smoothly discharging unfolded sheet/sheaf.

<<Ridge sensor **630**>>

FIG. **38** is a schematic perspective view illustrating a ridge sensor **630** which is provided for the accumulating tray unit **600** and FIG. **39** is a diagram illustrating the state in whish a weekly-magazine-like sheaf is stacked or stored.

The weekly-magazine-like sheaf is stacked on the accumulating tray **601** such that the bound section rises and is in the shape of a mountain. The accumulating tray unit **600** of the present embodiment particularly comprises a ridge sensor **630** which detects the ridge **633** of a weekly-magazine-like sheaf. The control of the elevation of the accumulating tray **601** is additionally attained based on the detection of the ridge **633** by the ridge sensor **630**.

The ridge sensor **630** is constructed of a transmission type photosensor provided with a light-emitting device **631** and a light-sensitive device **632**. The light-emitting device **631** and the light-sensitive device **632** are disposed so as to transverse obliquely the upper section of the accumulating tray **601** in the width direction and keep a certain distance ("L2" in FIG. **39**) downward from the nip part of the discharging rollers **122**, **123**. The ridge sensor **630**, therefore, is possessed of an optical axis which intersects the edge line of the ridge **633**. The ridge sensor **630** is also mounted on the supporting plate **634** (FIG. **33**). The distance L2 is such that the leading end of the weekly-magazine-like sheaf discharged by the discharging rollers **122**, **123** is enabled to pass over the ridge **633** of the weekly-magazine-like sheaf stored on the accumulating tray **601**. Specifically, the distance L2 is a size larger than the length of the leading end of the weekly-magazine-like sheaf which hangs down while being discharged.

The control of the elevation of the accumulating tray **601** based on the detection attained by the ridge sensor **630** is carried out as follows, in concert with the control of the forward and backward motion of the auxiliary guide **125**.

The accumulating tray **601** is lowered by actuating the drive motor even when the upper face sensor **606** has not detect the sheaf when the ridge sensor **630** continuously detects the ridge **633** of the sheaf for the duration of the time t [second] during a certain period following the detection of the trailing end of the sheaf by the discharged sheet sensor **124**, provided that the time t is shorter than the period. The downward movement of the accumulating tray **601** is stopped by halting the rotation of the drive motor when the detection of the ridge **633** by the ridge sensor **630** is interrupted (in case of a transmission state). The timer is reset, however, when the ridge sensor **630** fails to detect the ridge **633** continuously for the duration of the time t [second].

By forcibly lowering the accumulating tray **601** with respect to the presence of the ridge **633**, the upper most section (ridge **633**) of the weekly-magazine-like sheaf stacked on the accumulating tray **601** is always kept at a lower position separated by the distance L2 from the nip part of the discharging rollers **122**, **123**, irrespectively of the number of sheets stacked on the tray **601**. The weekly-magazine-like sheaf being discharged cannot contact to the weekly-magazine-like sheaf already stored on the accumulating tray **601**. The leading end of the sheaf being discharged infallibly falls on the further downstream side along the discharging direction from the ridge **633** of the sheaf on the accumulating tray **601**. In brief, the leading end of the sheaf avoids coming in the contact with the ridge **633** of the stored sheaf and does not cause the discharge failure of the sheaf.

The control of the elevation of the accumulating tray **601** based on the detection attained by the ridge sensor **630** is executed exclusively during the discharge of a weekly-magazine-like sheaf. The control of the elevation based on the detection attained by the upper face sensor **606** is executed during the discharge of other forms of sheet/sheaf. Therefore, the property of discharging such other forms of sheet/sheaf is retained perfectly.

<<Construction of control system>>

The system for controlling the various processing will be explained below. FIG. **40** is a block diagram of the control system for executing the various processing.

The control system is composed of a CPU **910** which controlling the copying machine, a CPU **950** which controls the ADF **12**, and a CPU **980** which controls the finisher **100**. These CPUs are provided respectively with ROM **911**, **951** and **981** which store the control programs and RAM **912**, **952** and **982** which function as relevant working areas.

The CPU **910** for the copying machine is provided with an image memory **825** which stores a scanned image data and an image data processing unit **820** which executes such image processing as rotation, enlargement, and reduction of the image based on the image data stored in the image memory **825**. A CCD line sensor **822** of the image reader is connected to the image data processing unit **820** through an A/D converter **821** which converts the scanned analog signal into a digital signal. Further, the image data processing unit **820** controls a laser device **832** of an image forming device (not shown) through a D/A converter **831** which converts a digital signal as a digital image data to an analog signal as an analog image data for outputting.

Various driven units and sensors are connected to the CPU **980** for the finisher for controlling and actuating the various units or devices of the finisher. The driven units include the stepping motors **128**, **210** and **408**, the DC motor **130**, the drive motors **127**, **304**, **406**, **512**, **515** and **604**, many solenoids and clutches, the switch claws **103**, **107** and **201**, etc. The sensors include the home position sensors **230**, **405** provided in the folding device **200** and the additional-work tray unit **400** respectively, the empty sensor **605**, the upper face sensor **606** and the ridge sensor **630** provided in the accumulating tray unit **600**, the sheet sensors **102**, **105**, **108**, **112**, **118**, **124**, **137** and **225** provided in the conveying paths for sheet/sheaf, the pulse disc sensors **407**, **432**, **513** and **552** for controlling the rotation of motors, and other sensors **410** and **516**. The ROM **981** connected to the CPU **980** for the finisher stores the certain distance "α" for calculating the moving length of the trailing end stopper **403** and the number of sheets as thresholds for determining leading end bind and training end bind.

The CPU **910** for the copying machine calculates the number of output sheets besides the basic operations proper for a copying machine (such as reading an image data on a document, storing the image data in memory, editing or processing the image data, forming an edited image on a paper, and outputting the paper). Specifically, the CPU **910** controls the document feeding of the ADF **12**, obtains the number of documents from the ADF **12**, and calculates the number of output sheets based on the number of documents and the copy mode inputted through the control panel. The result of the calculation is inputted to the CPU **980** for the finisher. The CPU **980** effects the choice between the leading

end bind and the trailing end bind based on the threshold value, a level of priority concerning the productivity, etc. In case of the trailing end bind, the CPU **980** inputs an instruction for rotating an image to the CPU **910** for the copying machine. In the above manner, the leading end bind or the trailing end bind is automatically selected. The user optionally makes the selection through the control panel besides the automatic selection of the leading end bind and the trailing end bind. In the case, it is automatically judged whether or not the output image is rotated, based on the binding position for a specified image and the instructed stapling position (leading end bind or trailing end bind). A command for rotating the output image and effecting right bind even in the case of trailing end bind is generated when the user instructs the right bind and the trailing end bind.

The entire disclosure of Japanese Patent Application No. 09-058120 filed on Mar. 12, 1997, including the specification, claims, drawings and summary are incorporated herein by reference in its entirety.

What is claimed is:

1. A finisher for stacking sheets comprising:

a receiving tray unit to receive a plurality of sheets;

a first regulating device to contact leading edges of a plurality of sheets received within the receiving tray unit;

a second regulating device, movable in a direction parallel to a direction of discharge of a sheet received in the receiving tray unit, to contact trailing edges of a plurality of sheets received within the receiving tray unit; and

a controller to control a movement of the second regulating device to push at least one sheet received within the receiving tray unit against the first regulating device to align a plurality of sheets received in the receiving tray unit in the direction of discharge.

2. A finisher according to claim **1**, wherein the plurality of sheets can include a folded sheet.

3. A finisher according to claim **2**, wherein the folded sheet is double-folded.

4. A finisher according to claim **2**, wherein the folded sheet has a z-folding.

5. A finisher according to claim **1**, wherein the finisher is adapted to receive a sheet from a connected device.

6. A finisher according to claim **5**, further comprising:

a folding apparatus for folding a sheet received from the device.

7. A finisher according to claim **5**, further comprising:

a stapling apparatus for stapling a plurality of sheets.

8. A finisher according to claim **5**, further comprising:

a binding apparatus for binding a plurality of sheets.

9. A finisher according to claim **1**, wherein, said second regulating device has a curved shape.

10. A finisher for stacking sheets comprising:

a receiving tray unit to receive a plurality of sheets;

a first regulating device to contact leading edges of a plurality of sheets received within the receiving tray unit;

a second regulating device, movable in a direction parallel to a direction of discharge of a sheet received in the receiving tray unit, to contact trailing edges of a plurality of sheets received within the receiving tray unit; and

a controller (i) to control a movement of the second regulating device to move the second regulating device to, and wait at, a position separated by a prescribed distance from at least one sheet received in the receiving tray unit and (ii) to move the second regulating device from the position to push at least one sheet received within the receiving tray unit to align a plurality of sheets received in the receiving tray unit in the direction of discharge.

11. A finisher according to claim **10**, wherein the plurality of sheets can include a folded sheet.

12. A finisher according to claim **11**, wherein the folded sheet is double-folded.

13. A finisher according to claim **11**, wherein the folded sheet has a z-folding.

14. A finisher according to claim **10**, wherein the finisher is adapted to receive a sheet from a connected device.

15. A finisher according to claim **14**, further comprising:

a folding apparatus for folding a plurality of sheets.

16. A finisher according to claim **14**, further comprising:

a stapling apparatus for stapling a plurality of sheets.

17. A finisher according to claim **14**, further comprising:

a binding apparatus for binding a plurality of sheets.

18. A finisher according to claim **10**, wherein, said second regulating device has a curved shape.

19. A method of alignment for a plurality of sheets stacked in a receiving tray unit of a finisher, the finisher having a first regulating device to contact leading edges of a plurality of sheets received within the receiving tray unit and a second regulating device, which is movable in a direction parallel to a direction of discharge of a sheet received in the receiving tray unit, to contact trailing edges of a plurality of sheets received within the receiving tray unit, the method comprising the steps of:

determining a length of a sheet to be received in the receiving tray unit during a finishing operation;

moving the second regulating device from a home position to a first position, wherein said first position is based on a determined length of a sheet to be received in the receiving tray; and

receiving a plurality of sheets into the receiving tray unit, wherein a trailing edge of each received sheet contacts the second regulating device as each sheet is received into the receiving tray unit.

20. A method according to claim **19**, further comprising the step of:

aligning a plurality of sheets received in the receiving tray unit in the direction of discharge,

wherein for the step of aligning, the second regulating device is moved in a reciprocating motion between the first position and a second position so as to push the plurality of sheets into the first regulating device.

21. A method according to claim **20**, wherein the aligning step is performed after all sheets in the plurality of sheets to be stacked have been received by the receiving tray unit.

22. A method according to claim **20**, wherein the aligning step is performed after each sheet in the plurality of sheets to be stacked has been received by the receiving tray unit.

23. A method according to claim **20**, wherein the aligning step is performed after a prescribed number of sheets in the plurality of sheets to be stacked has been received by the receiving tray unit.

* * * * *