



US005773729A

United States Patent [19]

[11] **Patent Number:** **5,773,729**

Nahar

[45] **Date of Patent:** **Jun. 30, 1998**

[54] **OVERLOAD PROTECTED LOADCELL**

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[21] Appl. No.: **718,621**

[22] Filed: **Sep. 17, 1996**

[51] **Int. Cl.**⁶ **G01L 1/00**

[52] **U.S. Cl.** **73/862.382; 73/862.634**

[58] **Field of Search** **73/862.382, 862.634, 73/862.639, 862.632, 862.627**

4,718,287	1/1988	Mishiborsky	73/802.634	X
4,744,709	5/1988	Hertel et al.	414/21	
4,836,036	6/1989	Jetter	73/862.65	
4,949,799	8/1990	Wernimont	177/211	
4,971,177	11/1990	Nojiri et al.	186/61	
5,052,505	10/1991	Naito et al.	73/862.634	X
5,139,100	8/1992	Brauneis	177/25.15	
5,183,125	2/1993	Schurr	177/211	
5,237,864	8/1993	Castle et al.	73/862.634	X

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[57] **ABSTRACT**

A loadcell for a weighing scale includes an elongate beam having mounting and free ends spaced apart along a beam axis to define a cantilever. The beam includes an upper surface spaced above the beam axis, and a lower surface spaced below the beam axis. A central aperture extends through the beam to define a plurality of spaced apart necks. The beam lower surface at the free end is spaced closer to the beam axis than at the mounting end to define a step. And, a plurality of strain gages are mounted to the beam adjacent to the necks for measuring strain thereat. Mounted in a scale housing, the beam free end abuts a housing mounting pad for limiting travel of the loadcell and providing overload protection therefor.

9 Claims, 2 Drawing Sheets

[56] **References Cited**

U.S. PATENT DOCUMENTS

4,107,985	8/1978	Sommer	73/862.382	X
4,143,727	3/1979	Jacobson	73/862.634	X
4,170,270	10/1979	Sette et al.	73/862.382	X
4,219,089	8/1980	Gard et al.	73/862.634	X
4,338,825	7/1982	Amlani et al.	73/862.382	
4,344,496	8/1982	Langlais et al.	73/862.634	X
4,438,823	3/1984	Hussels et al.	73/862.382	X
4,488,611	12/1984	Jacobson	177/211	
4,505,345	3/1985	Jetter	177/211	
4,632,198	12/1986	Uchimura	73/862.382	X
4,653,599	3/1987	Johnson	73/862.82	X

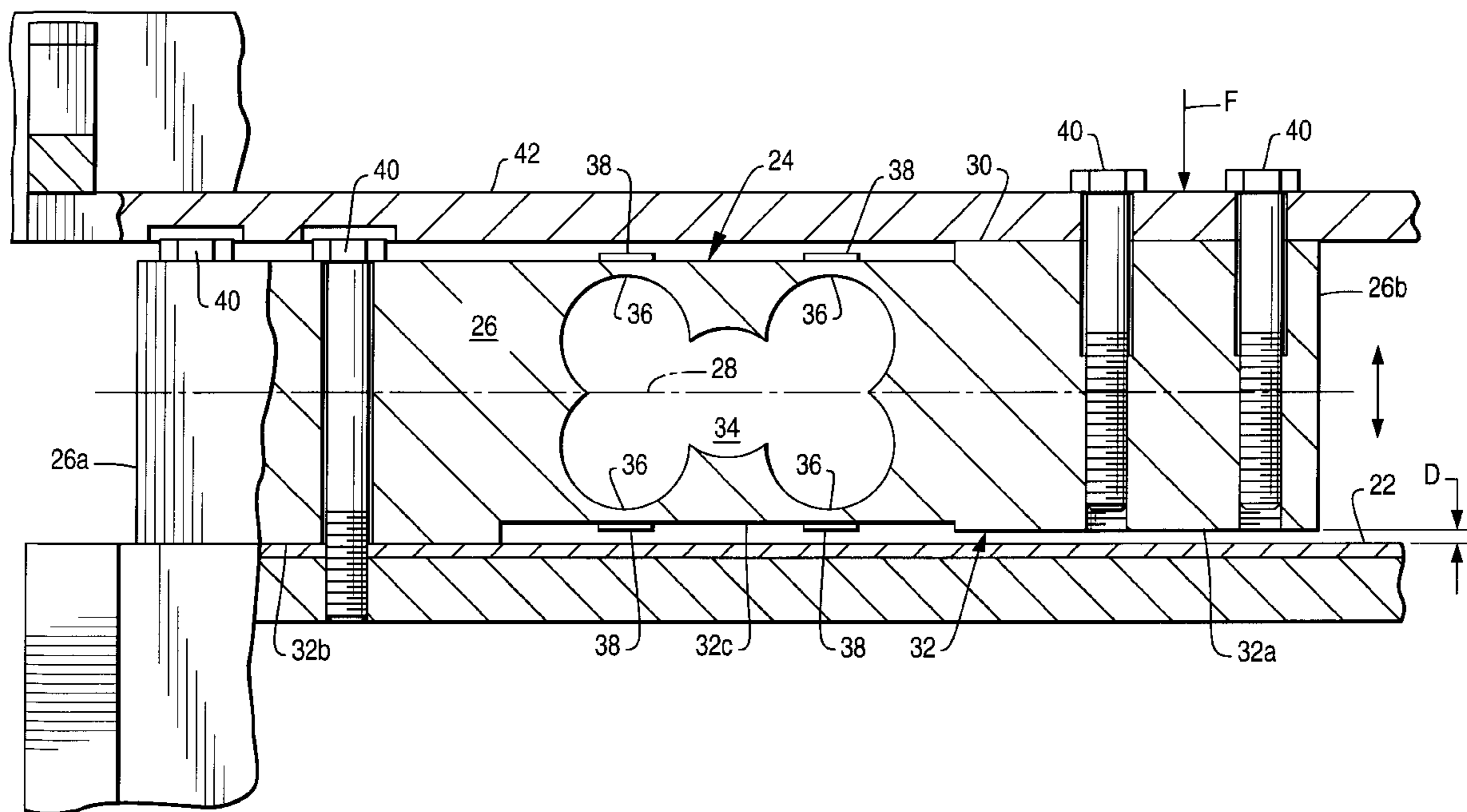


FIG. 1

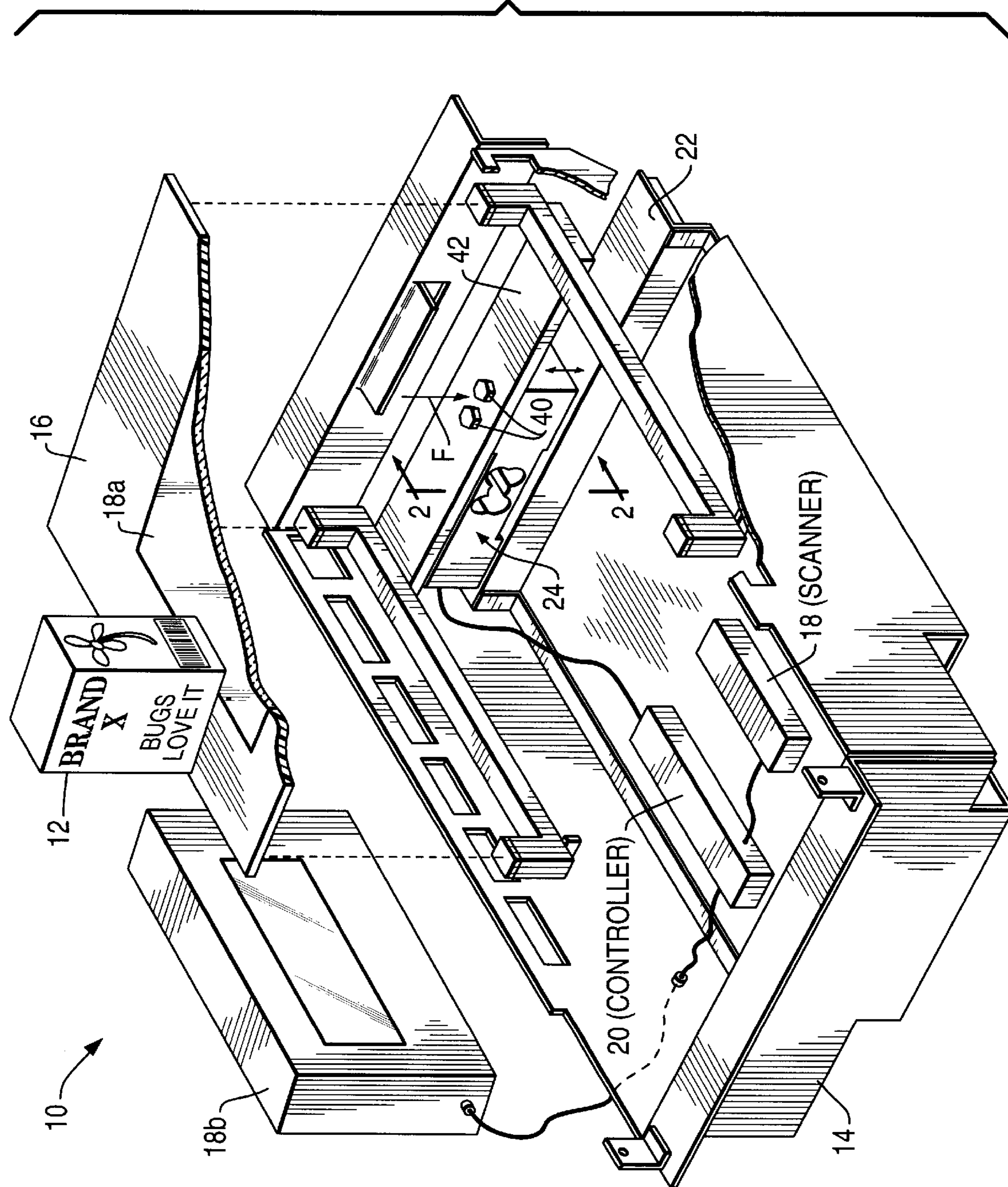
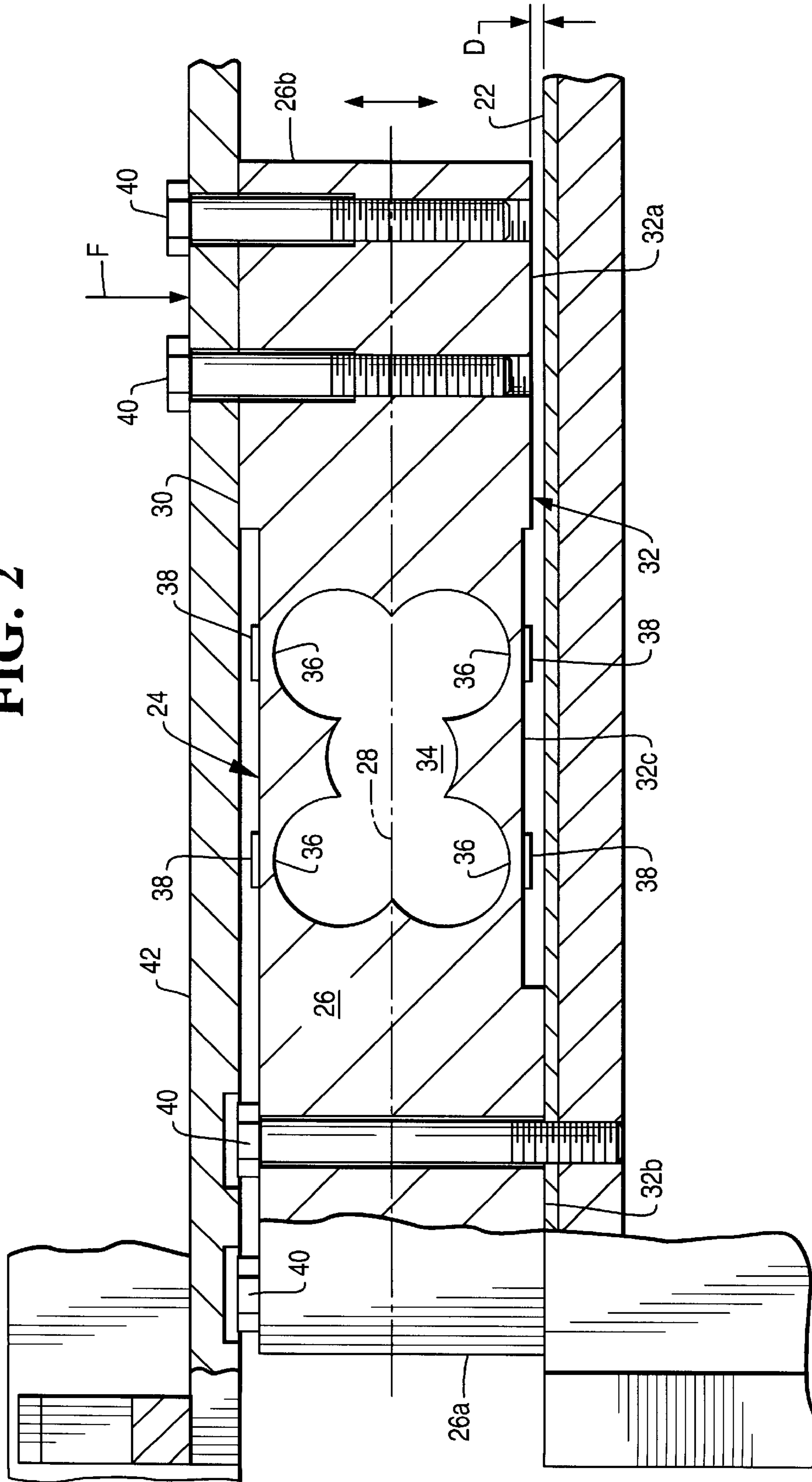


FIG. 2



OVERLOAD PROTECTED LOADCELL

BACKGROUND OF THE INVENTION

The present invention relates generally to weighing scales, and, more specifically, to a weighing scale loadcell having overload protection.

In one type of weighing scale, a loadcell in the form of an elongate beam is mounted as a cantilever, with a mounting end being fixedly mounted, and a free end supporting a scale plate on which articles may be placed for being weighed. The weight of the articles causes the loadcell beam to deflect and experience corresponding tension and compression bending stresses therein. The center of the beam includes a through aperture which maintains torsional rigidity of the beam and concentrates stress at local necks. Strain gages are suitably mounted at the necks for measuring strain which is directly proportional to the bending force exerted on the beam, which in turn may be used for determining the weight of the articles placed on the scale plate.

A typical scale has a specific weight limit, which if exceeded can damage the various components of the scale including the beam itself. Accordingly, the prior art includes various features for providing overload protection of the scale to prevent excessive bending of the loadcell. For example, suitable stops may be provided below the scale plate to prevent excessive movement thereof. Or, bending of the loadcell itself may be limited by a suitable stop such as an adjustable set screw.

More specifically, in one conventional weighing scale, a vertically extending set screw is mounted below the free end of the loadcell and is suitably adjusted to prevent the loadcell from deflecting beyond a predetermined limit. However, the allowed deflection of the loadcell during normal operation is extremely small over the weighing range of the scale. In one example, the scale may have a weighing limit of 50 pounds, with the deflection limit of the loadcell being about 9 mils. The set screw therefore requires very accurate adjustment and is then typically locked in adjustment by using a suitable adhesive. Since the 9 mil deflection limit is extremely small, just a few mils variation thereof out of adjustment can lead to undesirable permanent damage of the loadcell.

One advantage of the set screw, however, is that it effectively eliminates errors due to manufacturing tolerance and stack-up variations thereof. Without an adjustable set screw, the various joints between the loadcell and its mounting support must be accurately manufactured to extremely small tolerances so that when the loadcell is assembled in its housing, the resulting tolerance between the loadcell free end and an abutting structure is closely within the desired load limit. This is typically not practical, however, in view of the small limit of loadcell deflection on the order of 9 mils, especially when considering the goal of maintaining reduced cost of manufacture, assembly, and calibration.

SUMMARY OF THE INVENTION

A loadcell for a weighing scale includes an elongate beam having mounting and free ends spaced apart along a beam axis to define a cantilever. The beam includes an upper surface spaced above the beam axis, and a lower surface spaced below the beam axis. A central aperture extends through the beam to define a plurality of spaced apart necks. The beam lower surface at the free end is spaced closer to the beam axis than at the mounting end to define a step. And, a plurality of strain gages are mounted to the beam adjacent to the necks for measuring strain thereat. Mounted in a scale housing, the beam free end abuts a housing mounting pad for

limiting travel of the loadcell and providing overload protection therefor.

BRIEF DESCRIPTION OF THE DRAWINGS

The invention, in accordance with preferred and exemplary embodiments, together with further objects and advantages thereof, is more particularly described in the following detailed description taken in conjunction with the accompanying drawings in which:

FIG. 1 is an exploded, partly sectional, schematic view of an exemplary weighing scale including an overload protected loadcell therein.

FIG. 2 is an elevational, sectional view of the loadcell illustrated in FIG. 1 and taken along line 2—2.

DESCRIPTION OF THE PREFERRED EMBODIMENT(S)

Illustrated schematically in FIG. 1 is an exemplary embodiment of a weighing scale **10** configured for use in a typical retail establishment for weighing a consumer product **12**. In the exemplary configuration illustrated, the scale **10** is combined with a conventional laser bar code scanner for also conventionally reading the bar code found on the side of the product **12**.

More specifically, the scale **10** includes a suitable housing **14** in the form of a box with an open top in which are mounted various components of the scale assembly. Suitably mounted atop the housing **14** is a flat scale plate **16** on which the product **12** may be placed for the weighing thereof. Suitably mounted in part inside the housing **14** is a conventional laser bar code scanner **18** which cooperates with a horizontal window **18a** disposed coplanar in the scale plate **16**, and with a vertical window **18b** disposed adjacent to the scale plate **16** for projecting a laser beam across the product bar code and receiving back scattered light which is decoded in a conventional manner. A conventional controller **20** is operatively joined to the scanner **18** and to the scale **10** itself for controlling operation of both weighing the product **12** as well as reading the bar code label thereon.

The housing **14** further includes a flat and rigid mounting pad **22**, which in the exemplary embodiment illustrated in FIG. 1 is formed of relatively inexpensive, thin sheet metal suitably attached to a relatively rigid mounting bar integral in the housing **14** for providing a rigid base for the scale **10**. In accordance with the present invention, a loadcell **24** in the form of a cantilever is suitably joined to the mounting pad **22** and is operatively joined to the controller **20** for measuring strain induced by weight of the product **12**.

As shown in FIG. 2, the loadcell **24** includes an elongate cantilever beam **26** having a proximal, mounting end **26a** and an opposite distal, free end **26b** spaced longitudinally from the mounting end **26a** along a longitudinal centerline beam axis **28** to define the cantilever. The beam **26** also includes an upwardly facing, upper surface **30** extending the length of the beam **26** which is spaced above and generally parallel to the beam axis **28**. The beam **26** also includes a downwardly facing, lower surface **32** extending the length of the beam **26** which is generally parallel to and spaced below the beam axis **28**.

A central aperture **34** extends laterally through the beam **26** from side-to-side to define a plurality of spaced apart necks **36** of minimum cross section. A plurality of conventional strain gages **38** are suitably fixedly mounted, by an adhesive for example, to the beam **26** adjacent to respective ones of the necks **36** for measuring strain thereat in a

conventional manner. Each of the strain gages **38** is suitably electrically joined to the controller **20** shown in FIG. 1.

In accordance with the present invention, the beam lower surface **32** at the free end **26b** is spaced closer to the beam axis **28** than at the mounting end **26a** to define a step **32a** therebetween. The beam lower surface **32** at the mounting end **26a** defines a mounting land **32b**, and the step **32a** has a predetermined depth *D* measured inwardly toward the beam axis **28** relative to the mounting land **32b**.

As shown in FIG. 2, the mounting end **26a** of the loadcell **24** is suitably fixedly joined on the mounting pad **32b** by one or more removable threaded fasteners **40** which extend vertically through corresponding bores through the beam mounting end **26a** and threadingly engage the mounting pad **22**.

The beam free end **26b** up to the mounting land **32b** is preferably spaced above the mounting pad **22** by the step depth *D* for limiting deflection of the beam free end by abutment thereagainst for self-providing overload protection of the loadcell.

The fasteners **40** fixedly join the beam **26** to the housing **14** at the mounting pad **22**. The free end **26b** of the beam **26** in turn is suitably fixedly joined to a frame or platform **42** as shown in FIGS. 1 and 2. The platform **42** has a conventional H-bar configuration with a cross bar and two end bars having four corners upon which the scale plate **16** may simply rest. The cross bar of the platform **42** is preferably fixedly joined to the free end **26b** of the beam **26** by one or more of the threaded fasteners **40** which extend downwardly through holes in the platform **42** and threadingly engage corresponding threaded apertures in the beam distal end **26b**.

As shown in FIG. 1, the product **12** is placed atop the scale plate **16** for being weighed, with the weight thereof creating a downwardly directed force *F* which is exerted atop the beam free end **26b** as shown in more particularity in FIG. 2. The applied force *F* then causes the beam free end **26b** to deflect downward toward the mounting pad **22** thus bending the beam **26** in a cantilever fashion. The reaction bending loads are carried through the beam mounting end **26a** and into the mounting pad **22**.

The corresponding tension and compression bending stresses generated in the beam **26** are concentrated in the necks **36**, with the strain gages **38** positioned thereat measuring strain in a conventional manner. The measured strain is communicated to the controller **20** which is suitably calibrated for determining the weight of the product **12** corresponding with the specific amount of strain detected in a conventional manner. Accordingly, the scale plate **16** and platform **42** are freely supported atop the beam free end **26** so that a product **12** placed thereon correspondingly bends the beam **26** from which the weight of the product **12** may be determined.

By accurately positioning the step **32a** of the beam lower surface **32** relative to the mounting land **32b** thereof, self-overload protection of the beam **26** may be effected upon abutment of the free end **26b** and the mounting pad **22** when downward travel reaches the limit of the provided clearance therebetween.

This overload protection of the loadcell **10** is effected without a conventional adjustable set screw, or without multi-components which each require accurate manufacturing tolerance for providing overload protection. Overload protection is readily provided within the beam **26** itself by the specifically configured beam lower surface **32**. As shown in FIG. 2, the mounting pad **22** is flat preferably along its entire length under the beam **26**, and may be readily pro-

vided by using flat sheet metal which may be thick enough to provide its own rigidity, or may be relatively thin and supported upon a rigid bar as illustrated.

The beam **26** is mounted on the mounting pad **22** so that the beam axis **28** is spaced parallel to the mounting pad **22**. The mounting land **32b** is preferably flat and directly abuts the mounting pad **22**, with the pair of fasteners **40** providing a fixed and rigid joint thereat. The step **32a** is also preferably flat and parallel to the mounting land **32b**, and is initially spaced above the mounting pad by the depth *D* of the step **32a**.

In this arrangement, the step **32a** is accurately positioned relative to the mounting land **32b** and may be readily manufactured in a common machining operation using the mounting land **32b** as a reference from which the step **32a** may be accurately formed by suitable machining of metal therefrom with a single machining tolerance therefor. In this way, once the beam **26** is fastened to the flat mounting pad **22**, the initial clearance *D* is accurately provided automatically without the need for any adjustment, such as that provided by suitable shims, and without stack-up tolerances which would alter the desired initial clearance. For example, the initial depth *D* of the step **32a** and the corresponding clearance provided thereby in an exemplary embodiment is about 9 mils which is a very small clearance. This very small clearance is accurately created automatically by mounting the pre-machined beam **26** on the mounting pad **22**.

In the preferred embodiment illustrated in FIG. 2, the depth *D* of the step **32a** is sized sufficiently small so that cantilever deflection by bending of the beam free end **26b** relative to the mounting end **26a** over the depth *D* does not exceed about two percent (2%) yield stress of the beam **26** itself. The beam free end **26b** may be displaced downwardly only until it abuts the oppositely facing mounting pad **22** which prevents further downward travel thereof for providing overload protection of the loadcell **10**. In an exemplary embodiment for a metal beam **26**, the initial step depth *D* of about 9 mils ensures that the bending stress within the beam **26** at the corresponding necks **36** does not exceed the 2% yield stress of the beam material.

In the exemplary embodiment illustrated in FIG. 2, the mounting land **32b** only extends over the region of the pair of fasteners **40** for providing an effective compression joint thereat. The mounting land **32b** extends to about the central aperture **34**. And, the step **32a** is provided along the beam free end **26b** to at least about the central aperture **34** for ensuring that the beam **26** may deflect without obstruction until the overload limit is reached upon abutment of the step **32a** and the mounting pad **22**.

In the exemplary embodiment illustrated in FIG. 2, the beam **26** further includes a central recess **32c** extending along the lower surface **32** between the mounting land **32b** and the step **32a** for receiving at least one of the strain gages **38**, and in the exemplary embodiment both of the lower strain gages **38**. The strain gages **38** may be suitably covered with a protective coating without interfering with the free bending travel of the beam **26** and step **32a**. The step **32a** in effect extends from the distal-most end of the beam **26** up to the mounting land **32b** for preventing obstruction between the beam **26** and the mounting pad **22** until the step **32a** reaches its overload travel limit in abutment against the pad **22**.

The central aperture **34** may take any suitable configuration along with the necks **36** to provide suitable torsional rigidity of the beam **26** with suitable stress concentration at the necks **36** so that the strain gages **38** may accurately

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measure strain to determine the corresponding weight of the product 12. As shown in FIG. 2, the beam central aperture 34 may be defined by two pairs of intersecting corner holes symmetrically intersecting a center hole along the beam axis 28 for defining four identical necks 36 between the corner holes and the beam upper and lower surfaces 30, 32. Two of the strain gages 38 are mounted on the beam upper surface 30 adjacent to the corresponding necks 36, and two of the strain gages 38 are mounted on the beam lower surface recess 32c opposite the corresponding necks 36.

As disclosed above, the overload protection clearance represented by the step 32a is built into the lower surface of the beam 26 itself and thereby eliminates the conventional set screw and required adjustment thereof. Substantially small overload clearances on the order of about 9 mils may be accurately implemented and are necessarily permanent without the need for field adjustment. The overload clearance is automatically effected by simply fastening the beam mounting end 26a directly to the mounting pad 22 which automatically positions the step 32a at the required spacing from the opposing portion of the mounting pad 22. The mounting pad 22 itself does not require any accurate machining of a cooperating abutting portion which would lead to manufacturing tolerance stack up problems. The mounting pad 22 may be readily manufactured by providing a suitably flat plate such as formed sheet metal of suitable thickness with or without a rigid backing plate. Assembly cost of the scale 10 will therefore be lowered, and field service of the scale 10 will be significantly easier since adjustment of the loadcell is no longer required.

Although the invention has been described in the environment of a combination scale and laser scanner, it may be used alone in any scale application in which a cantilever loadcell provides the weighing mechanism. The loadcell is simply fastened to a suitable mounting pad at its mounting end, with the free end having the corresponding inward step for providing unobstructed bending travel of the loadcell within its operating range until the free end of the loadcell bottoms out on the mounting pad upon exceeding the maximum weight capability of the loadcell. The loadcell is thereby protected against any further bending thereof which would overload or otherwise damage the loadcell itself.

While there have been described herein what are considered to be preferred and exemplary embodiments of the present invention, other modifications of the invention shall be apparent to those skilled in the art from the teachings herein, and it is, therefore, desired to be secured in the appended claims all such modifications as fall within the true spirit and scope of the invention.

Accordingly, what is desired to be secured by Letters Patent of the United States is the invention as defined and differentiated in the following claims:

I claim:

1. A loadcell for a weighing scale comprising:
an elongate beam including:
mounting and free ends spaced apart along a longitudinal centerline beam axis to define a cantilever;

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an upper surface spaced above said beam axis;
a lower surface spaced below said beam axis;
a central aperture extending laterally through said beam to define a plurality of spaced apart necks; and
said beam lower surface at said free end being spaced closer to said beam axis than at said mounting end to define a step therebetween; and

a plurality of strain gages mounted to said beam adjacent to respective ones of said necks for measuring strain thereat; and

wherein said beam lower surface at said mounting end defines a mounting land, and said step has a depth measured inwardly therefrom, and said step depth is sized so that cantilever deflection of said beam free end relative to said mounting end over said depth does not exceed about two percent yield stress of said beam.

2. A loadcell according to claim 1 wherein said step and mounting land extend to about said central aperture.

3. A loadcell according to claim 1 in combination with said scale wherein said scale further includes a housing having a mounting pad, and said loadcell mounting end is fixedly joined directly to said mounting pad, with said free end being spaced from said mounting pad by said step depth for limiting deflection of said beam by abutment thereagainst.

4. An apparatus according to claim 3 wherein:

said mounting pad is flat and spaced parallel to said beam axis;

said mounting land is flat and abuts said mounting pad; and

said step is flat and parallel to said mounting land, and is spaced above said mounting pad by said step depth.

5. An apparatus according to claim 4 further comprising a removable fastener extending through said beam mounting end and into said mounting pad for fixedly joining said beam to said housing.

6. An apparatus according to claim 5 further comprising a platform fixedly joined atop said beam free end for supporting a removable scale plate to weigh an article placed thereon.

7. An apparatus according to claim 6 wherein said step and mounting land extend to about said central aperture.

8. An apparatus according to claim 7 wherein said beam further comprises a recess extending along said lower surface between said mounting land and said step for receiving at least one of said strain gages.

9. An apparatus according to claim 8 wherein:

said beam central aperture is defined by two pairs of intersecting corner holes symmetrically intersecting a center hole for defining four of said necks between said corner holes and said beam upper and lower surfaces; and

a respective one of said strain gages is mounted on said beam upper and lower surfaces at said necks.

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