

[54] **ROTATIONAL VISCOMETER**  
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[51] Int. Cl.<sup>3</sup> ..... **G01N 11/14**  
[52] U.S. Cl. .... **73/60**  
[58] Field of Search ..... **73/54, 59, 60**

[56] **References Cited**

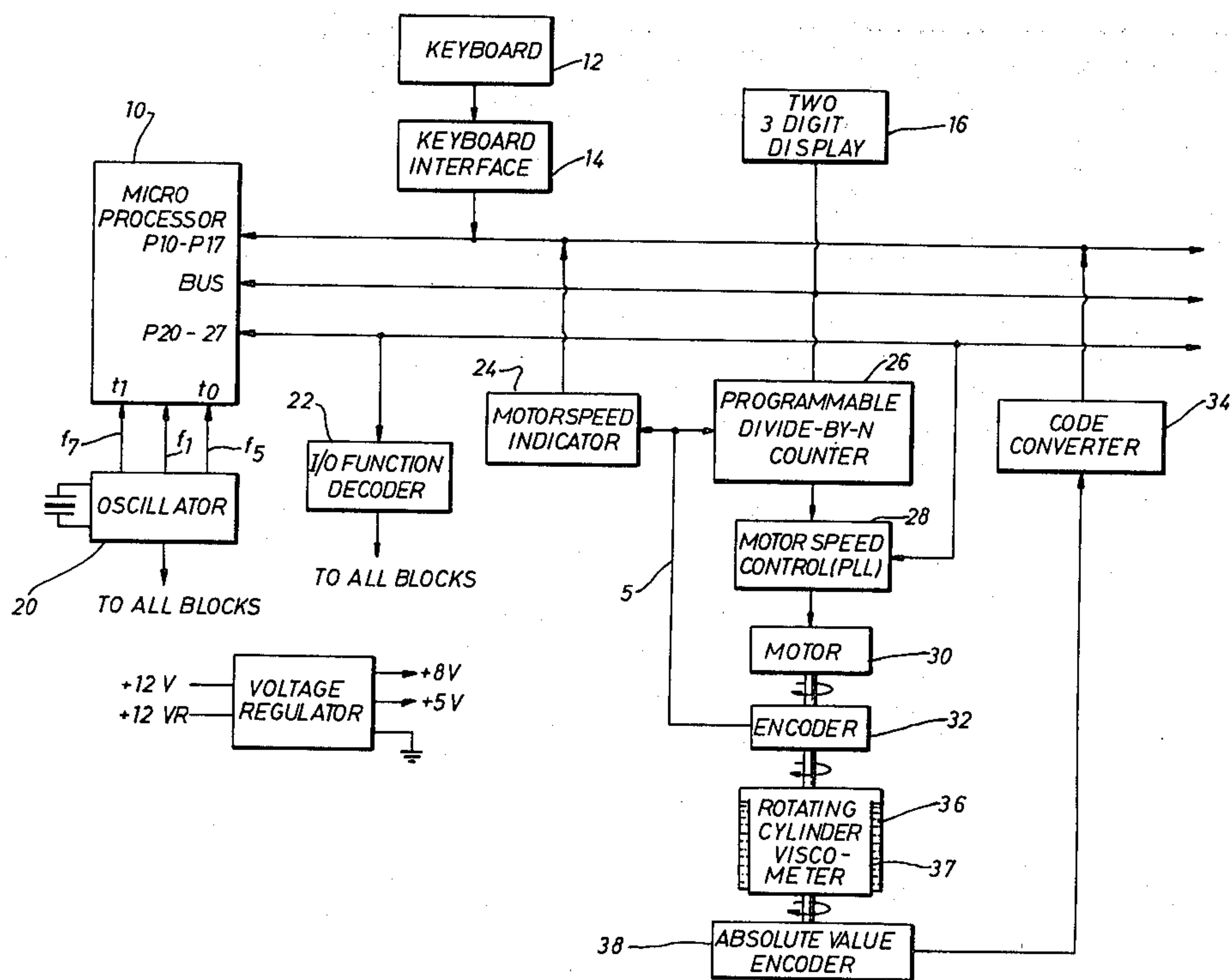
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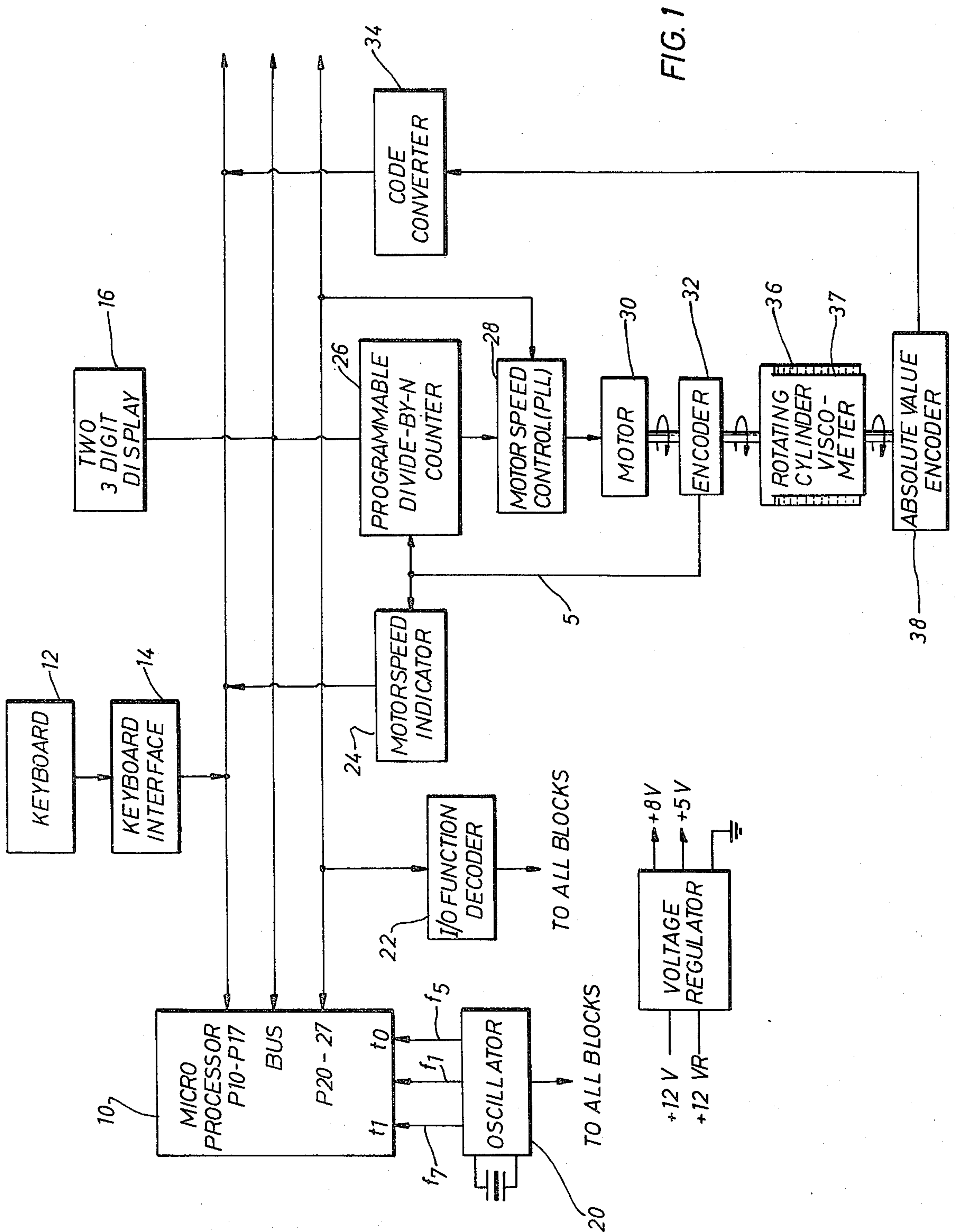
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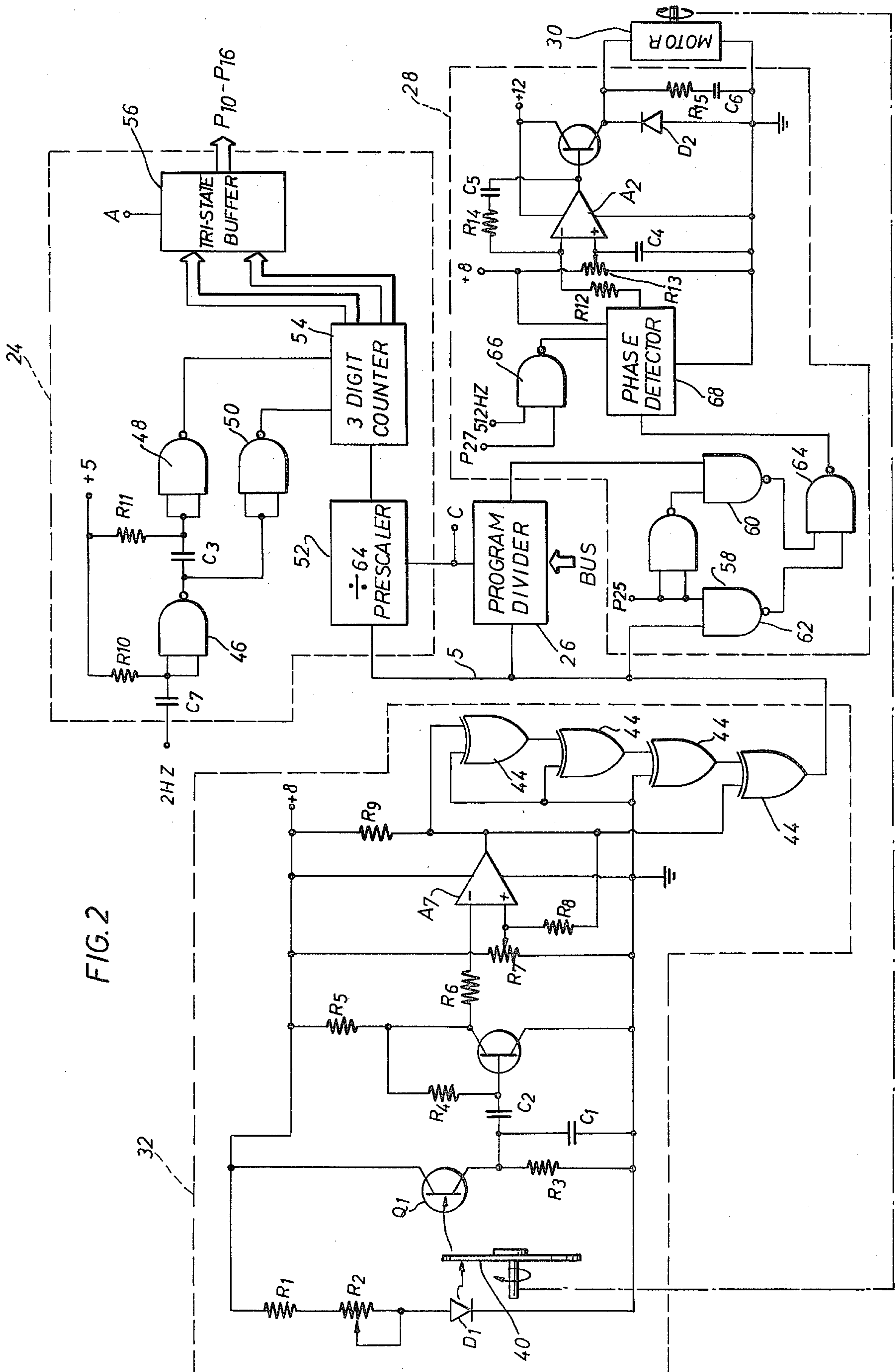
Primary Examiner—James J. Gill  
Attorney, Agent, or Firm—John N. Hazelwood; Peggy L. Smith

[57] **ABSTRACT**  
A microprocessor controlled rotating-cylinder viscometer for measuring the shear stress at a given shear rate of a fluid is disclosed. The fluid is contained between a rotatable outer cylinder and a rotatable inner cylinder. The inner cylinder rotates in response to the torque produced by rotation of the outer cylinder in the fluid. The amount of rotation of the inner cylinder is related to the shear stress of the fluid. A fast slewing accurate motor speed control is provided for maintaining rotation of the outer cylinder at the selected shear rates. An absolute value shaft encoder monitors the amount of rotation of the inner cylinder to obtain the shear stress readings. The steady state value of the shear stress is obtained by averaging a predetermined number of measurements taken over a sample interval to obtain a stress reading. Consecutive readings are compared until the difference between readings is less than  $\pm 1^\circ$ , the current shear stress then taken on the steady state value.

32 Claims, 16 Drawing Figures









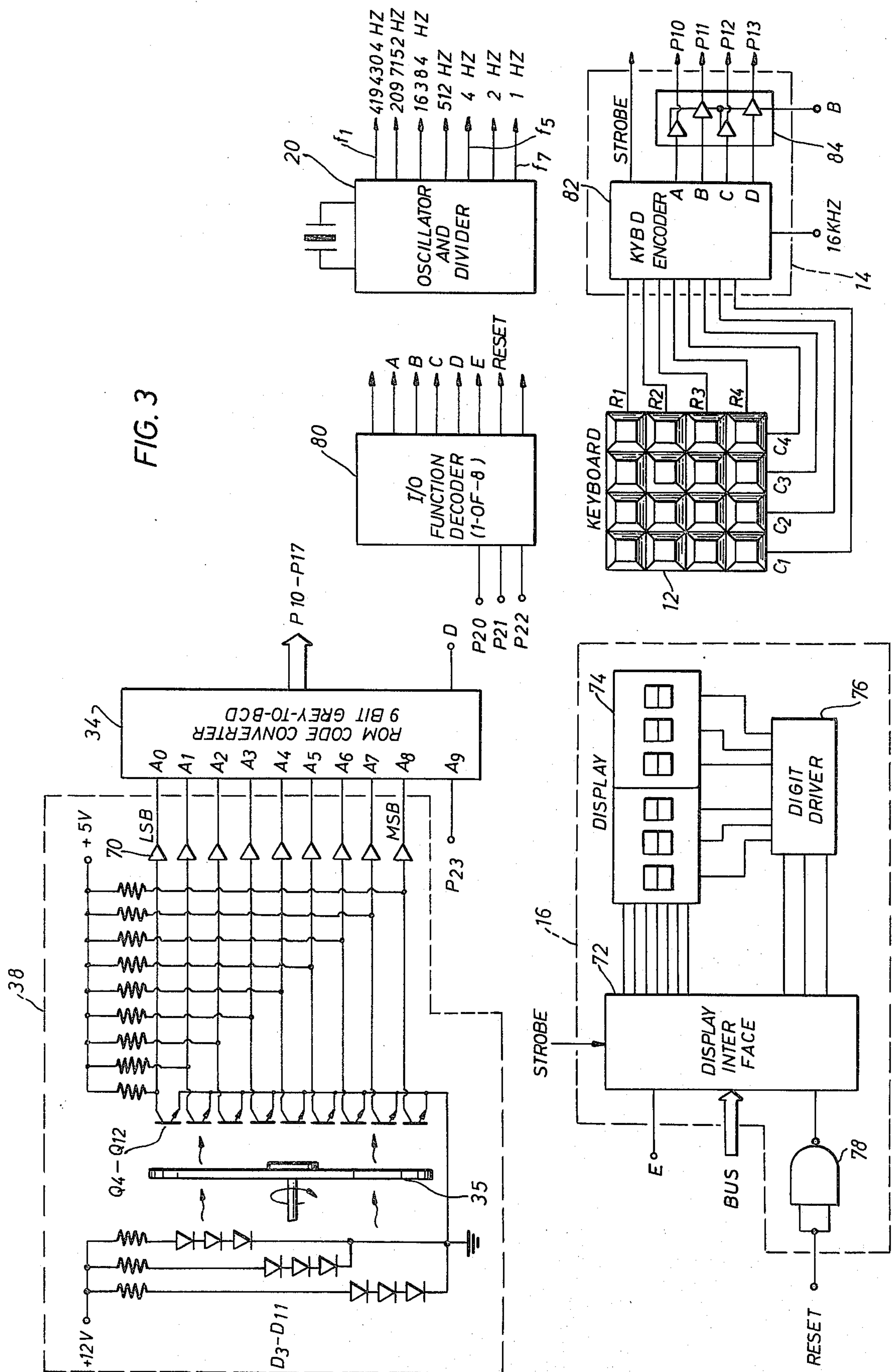


FIG. 4

MEMORY AND I/O ASSIGNMENTS

ROM

ADDRESS	ALLOCATION
000-3CF	PROGRAM MEMORY
3D0-3FF	BCD-7SEG. LOOKUP TABLE
400-7FF	PROGRAM MEMORY

RAM

ADDRESS	ALLOCATION
00-07	REGISTERS 0 TO 7
08-17	8 LEVEL STACK
18-1A	KEYBOARD DATA
1B-1D	RPM AND TIME DATA
1E-20	STRESS AND MESSAGE DATA
21-3E	STRESS STORAGE
40-5D	SPEED STORAGE
5E-7F	SCRATCH PAD MEMORY

P1

7	6	5	4	3	2	1	0
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P10-13 BCD INPUTS

P14-17 BCD AND DIGIT SELECT INPUTS

P2

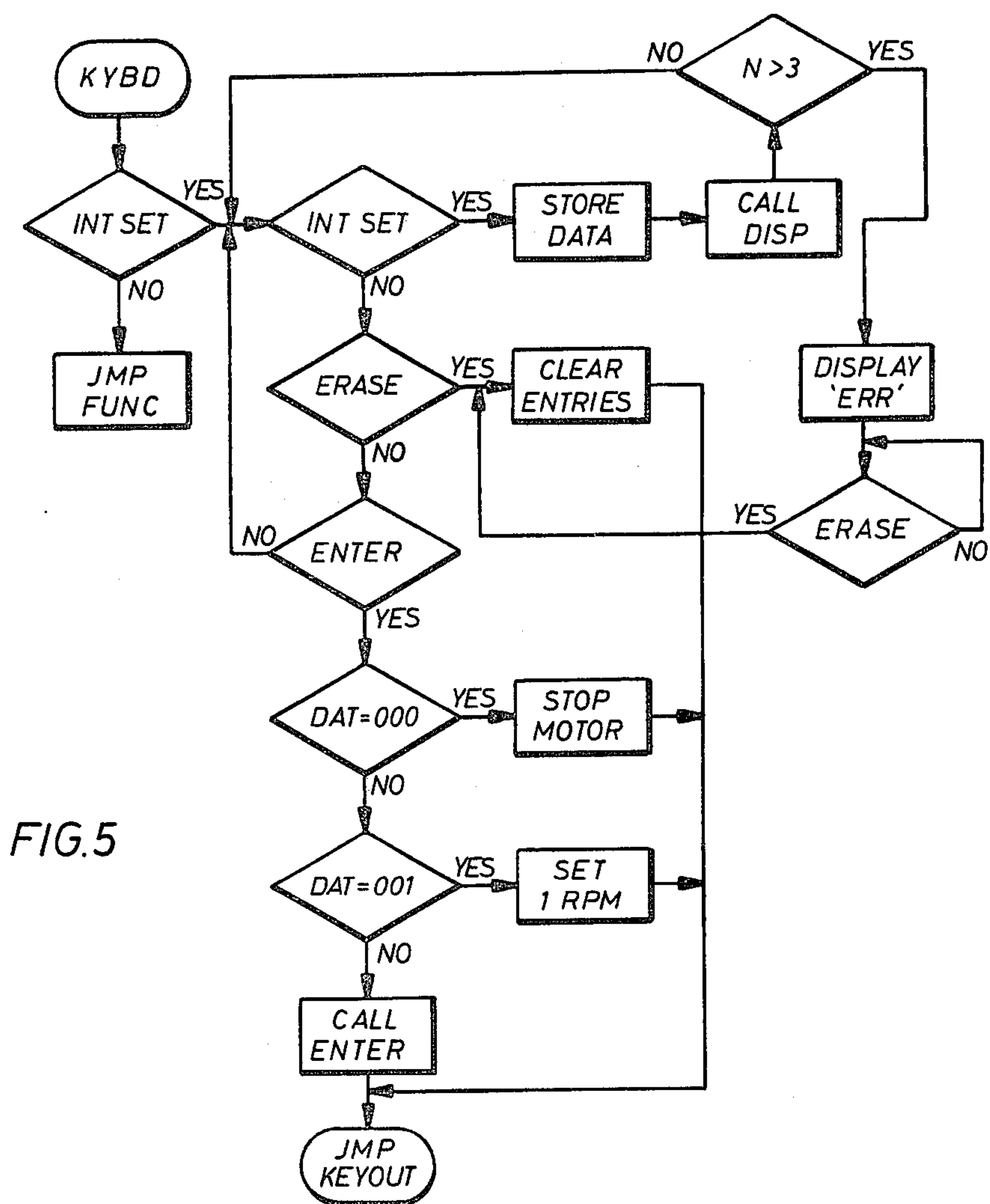
7	6	5	4	3	2	1	0
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P20-22 CHIP SELECT DECODER ADDRESS

P23-24 CONTROL OUTPUTS

P25 1RPM ACTIVATOR

P27 MOTOR ENABLE ACTIVATOR



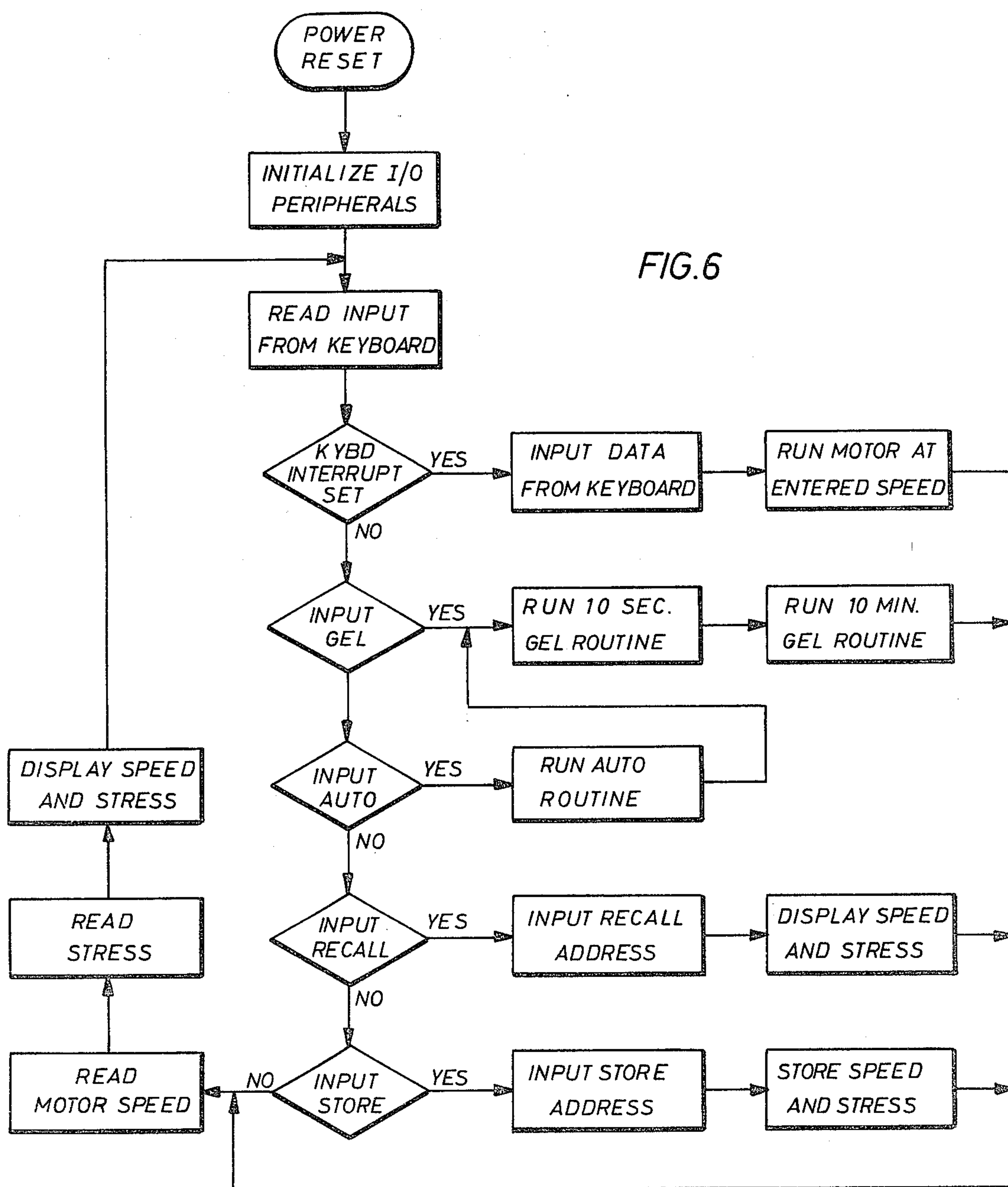


FIG. 7

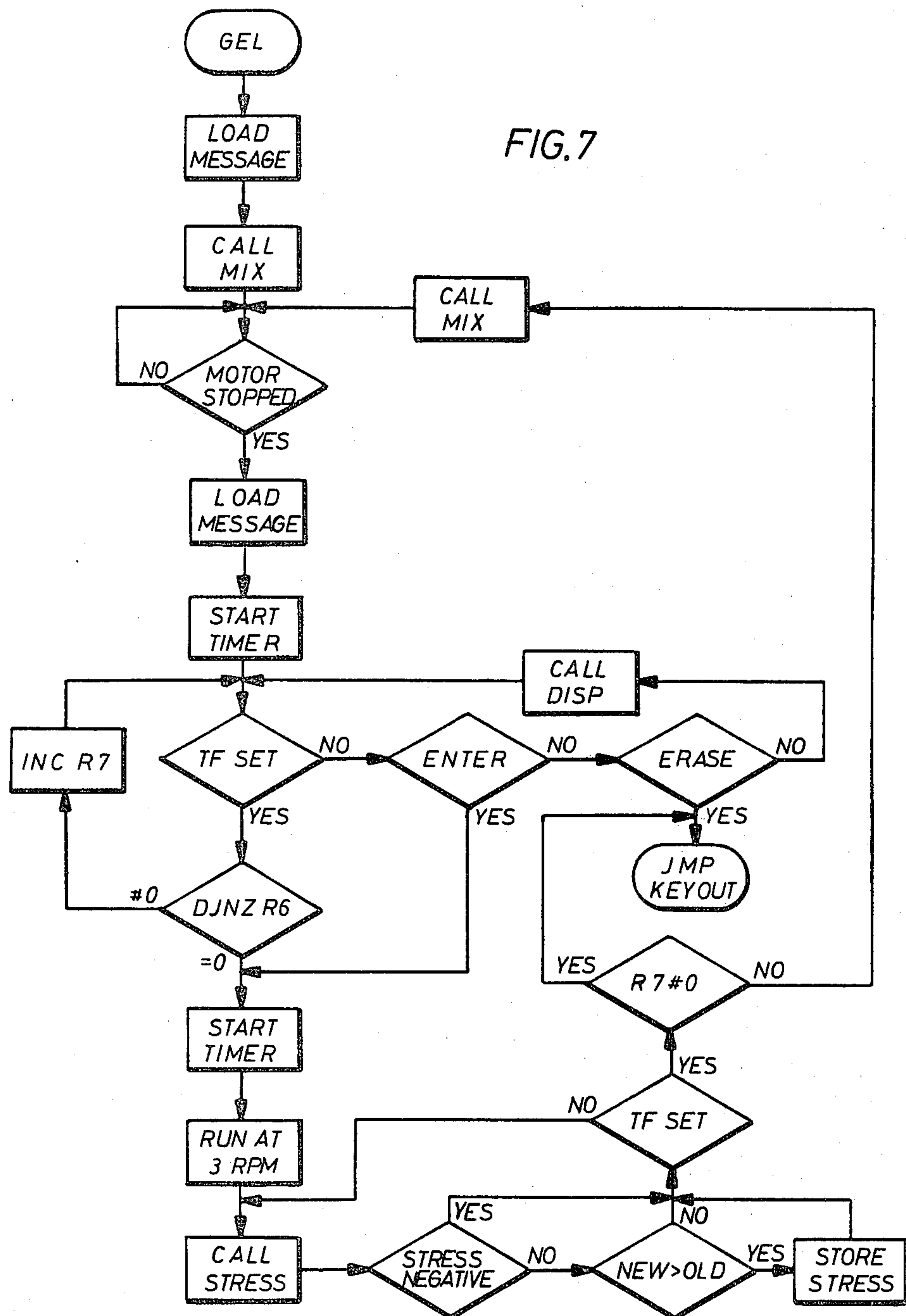
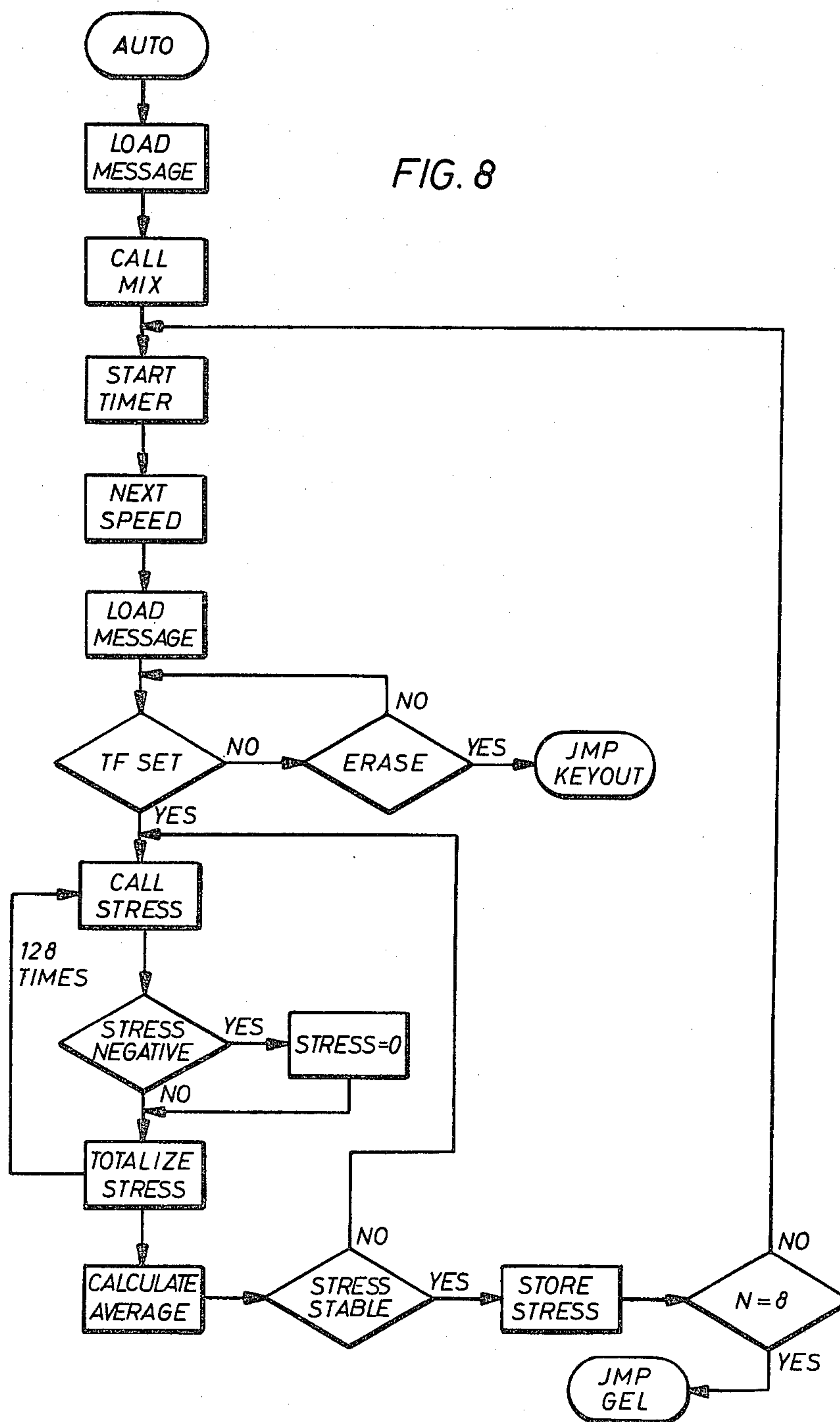




FIG. 8



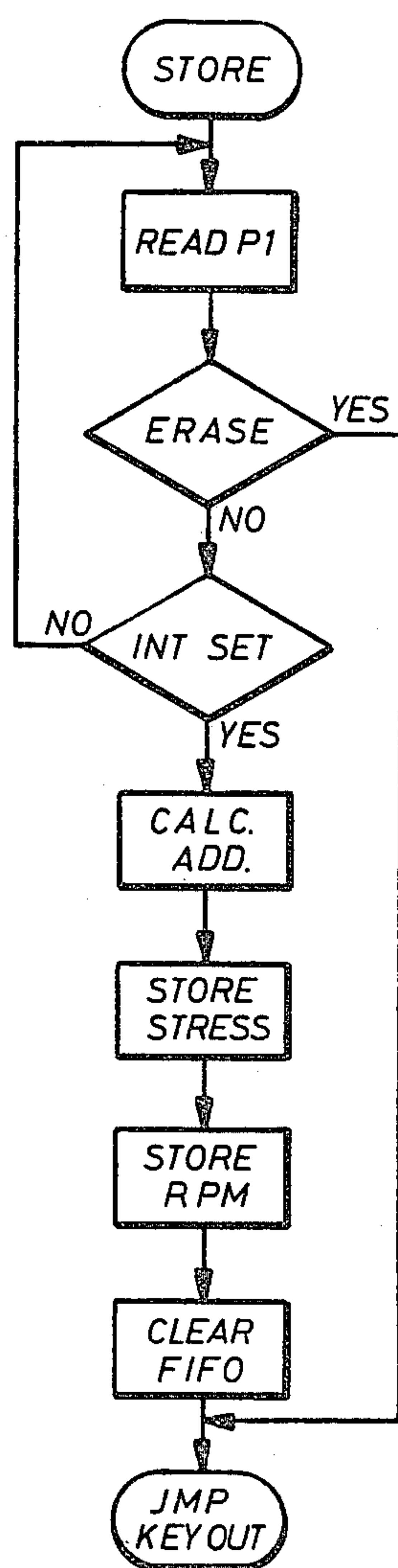


FIG. 9

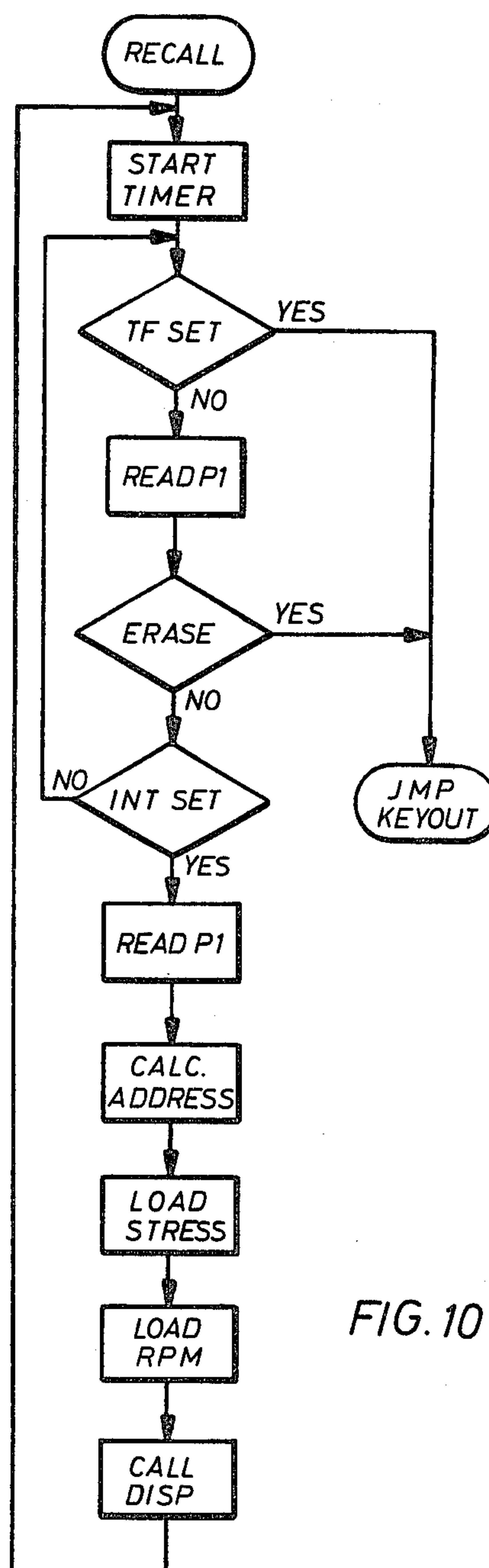


FIG. 10

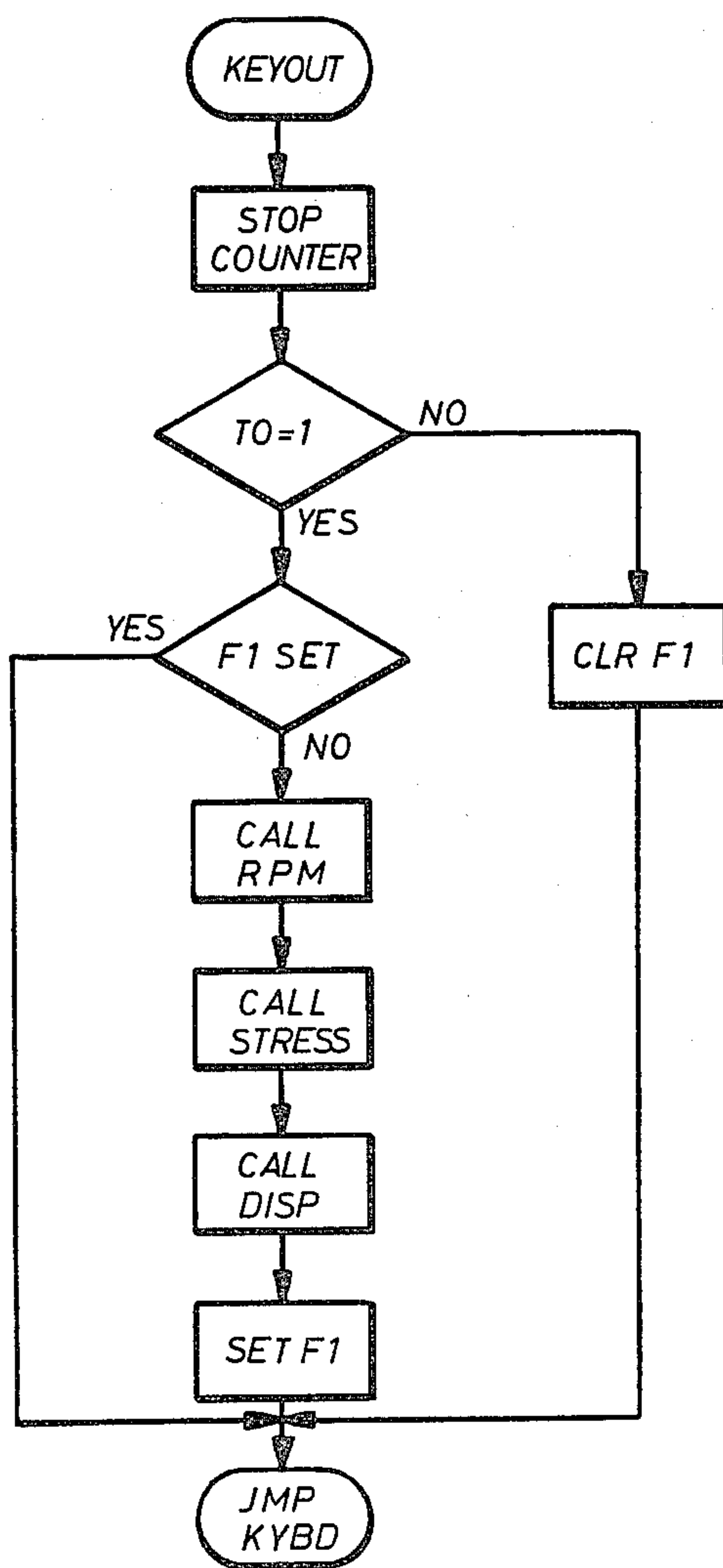


FIG. 11

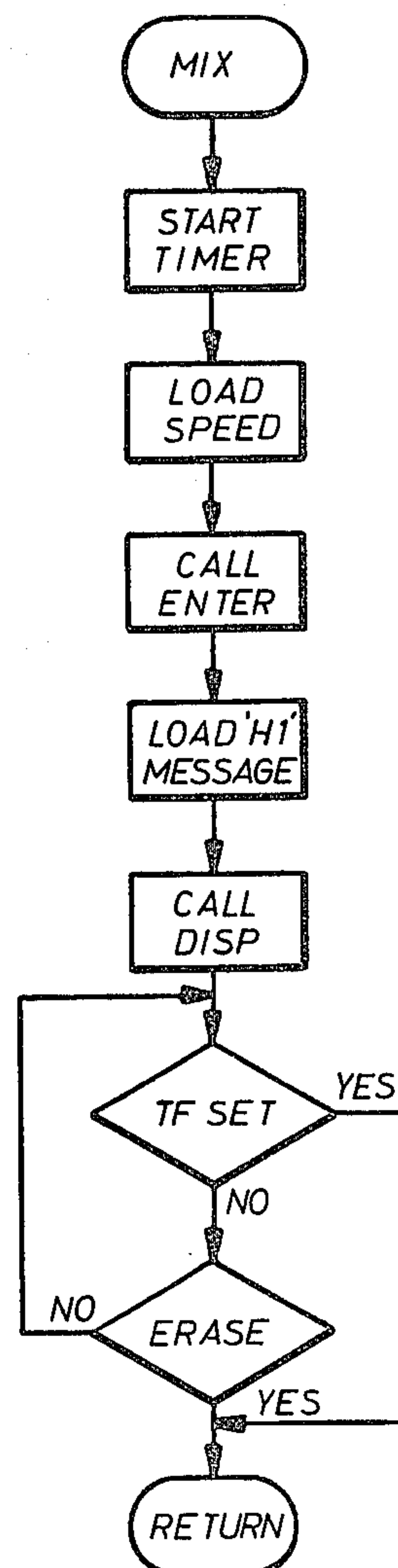


FIG. 12

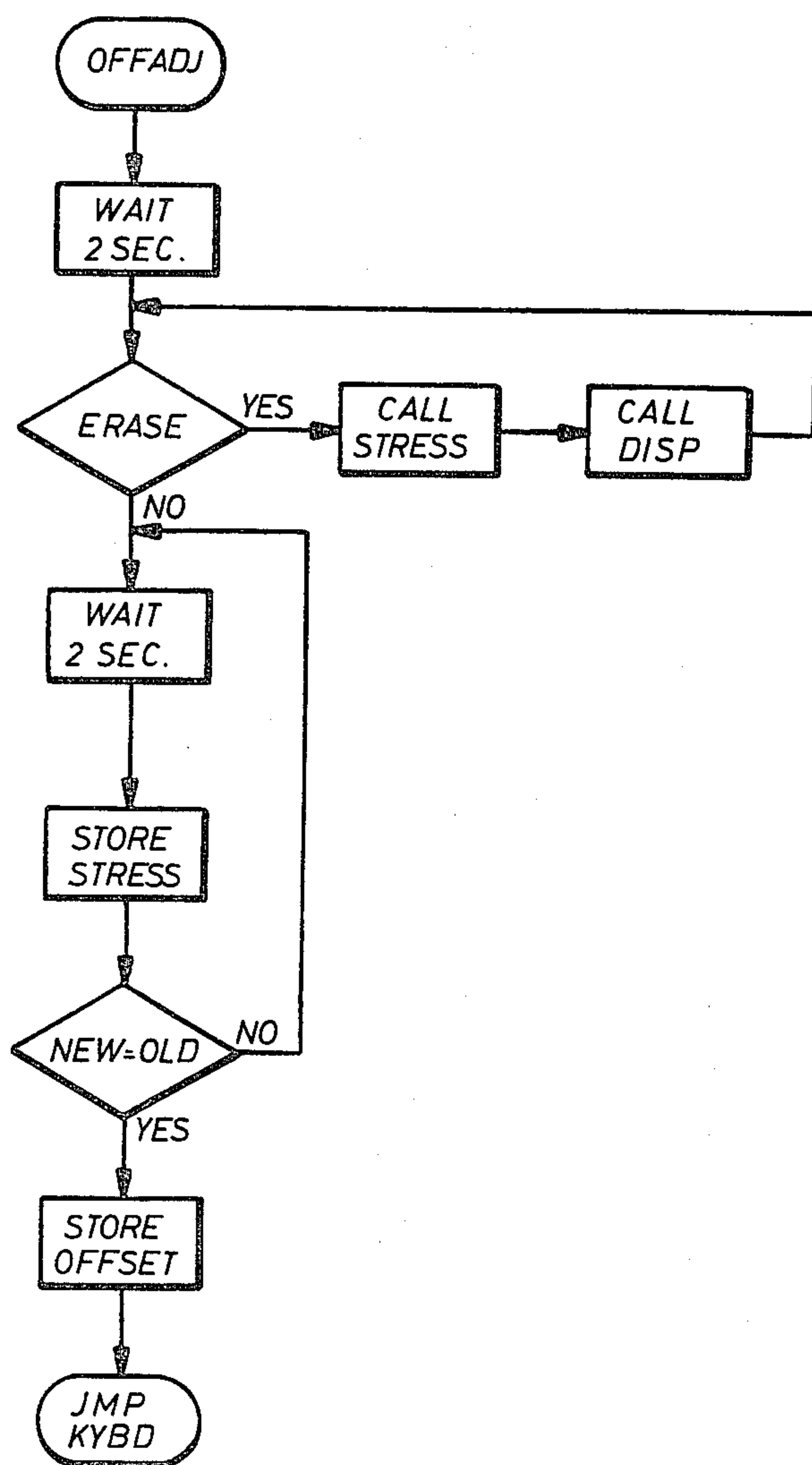


FIG.13

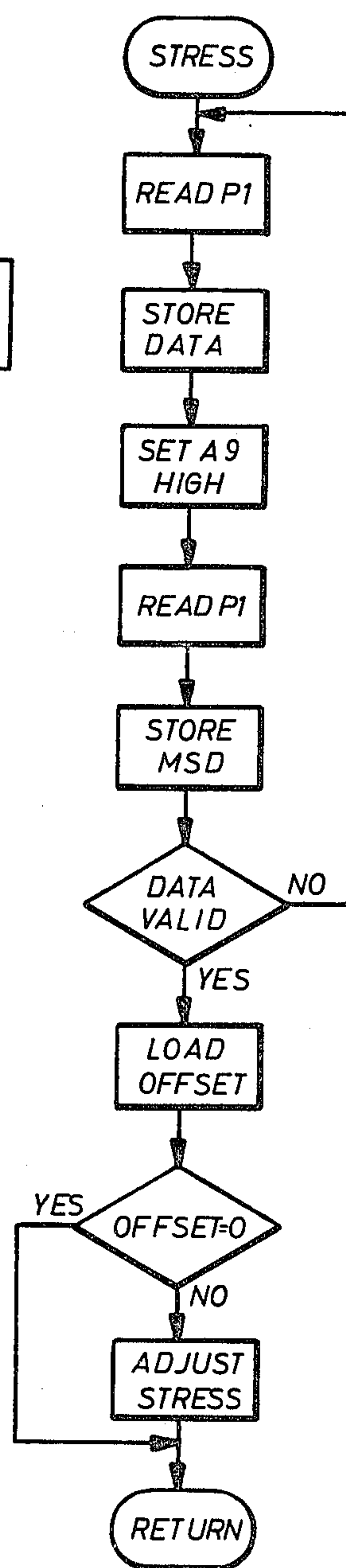


FIG.14



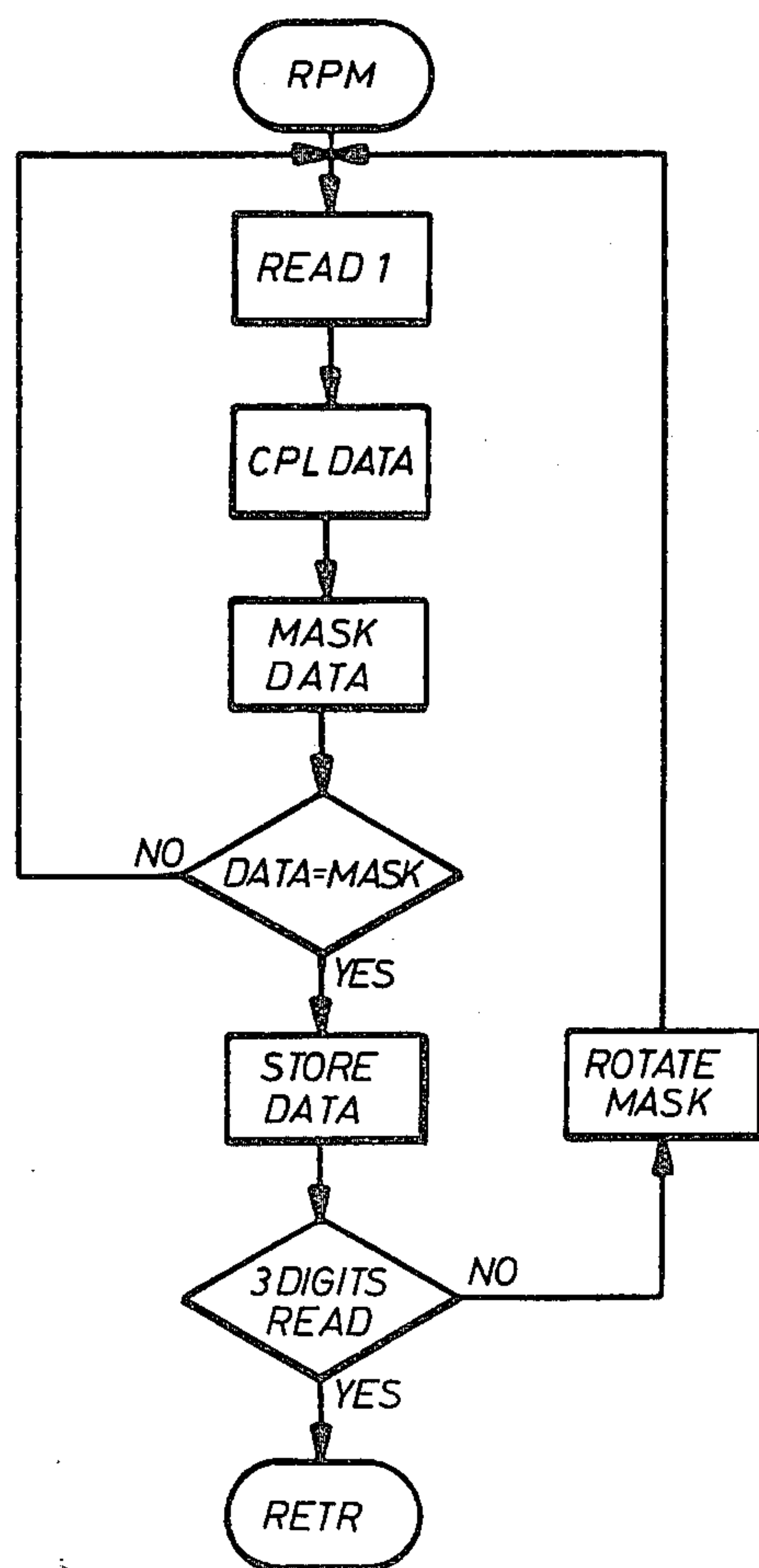


FIG. 15

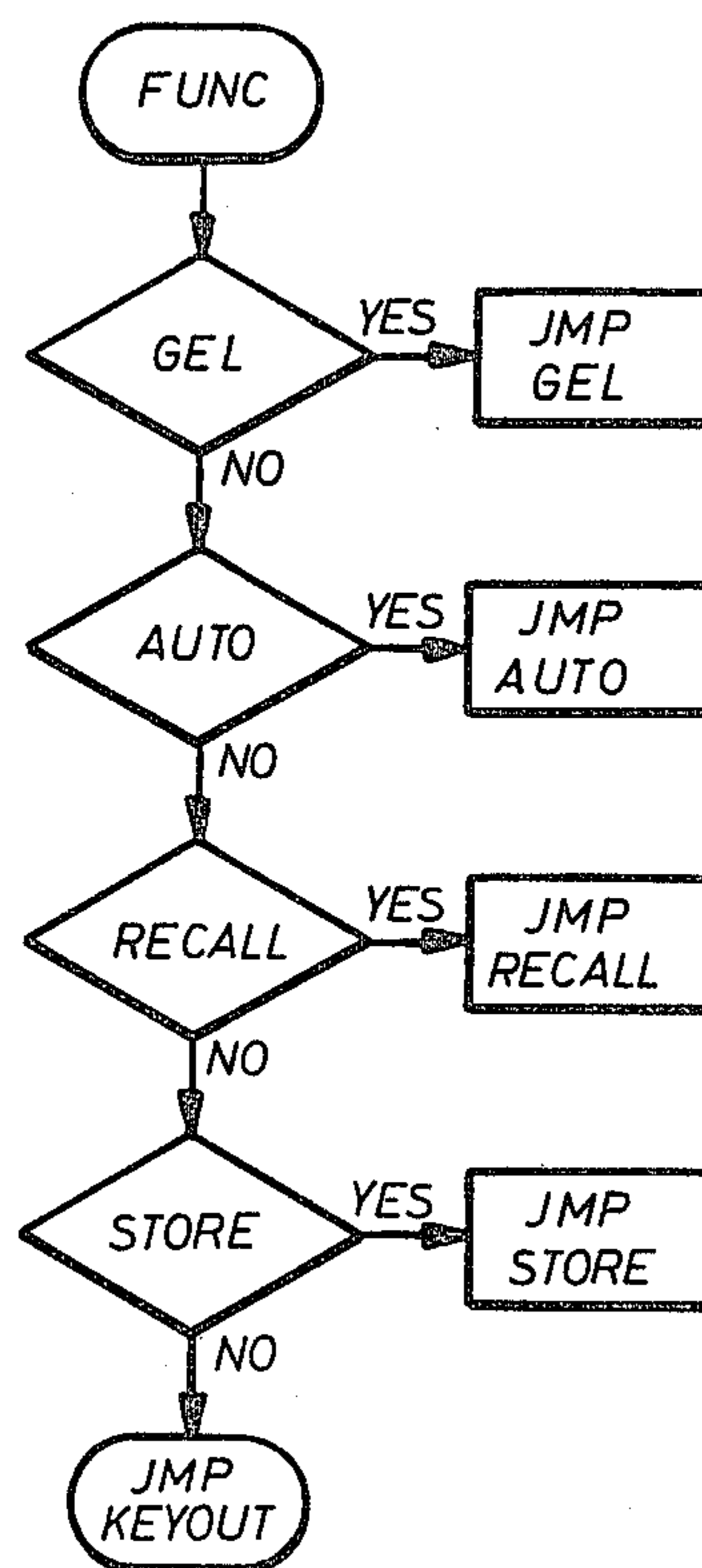


FIG. 16



## ROTATIONAL VISCOMETER

### BACKGROUND OF THE INVENTION

This invention relates to rotational viscometers for measuring rheometric properties of a fluid. More particularly, this invention relates to a microprocessor controlled rotational type viscometer for automatically and accurately obtaining the steady state shear stress of a fluid at various preselected shear rates.

Properties of fluids, such as the shear stress, shear strength, yield stress, plastic viscosity, etc., are important in many different industries. For example, viscometers are widely used in the drilling industry to measure these properties of drilling fluids that are used to drill oil and gas wells. Information obtained with viscometers is important in controlling the effectiveness of the drilling fluid in, (1) removal of cuttings from the bottom of the hole and carrying them to the surface, (2) holding the cuttings and weight material in suspension when circulation is interrupted, (3) releasing the cuttings and any entrained gases at the surface, (4) transmission of hydraulic horsepower to the drill bit, (5) minimizing annular pressure drops so as to avoid fracturing and the resulting loss of circulation in the uncased hole, (6) maximizing bore hole stability by controlling erosional effects on the well bore while circulating, and (7) reducing to a minimum any adverse effects upon the formation adjacent to the bore hole.

Direct-indicating concentric cylinder rotational viscometers powered by means of an electric motor or hand crank have found wide acceptance in the petroleum industry as an acceptable approach to measuring the viscosity of drilling fluid. In such a viscometer, the drilling mud is contained in the annular space between two cylinders. The outer cylinder or rotor sleeve is driven at a constant rotational velocity or shear rate. Located within the outer cylinder is an inner cylinder. The rotation of the outer cylinder in the mud produces a torque on the inner cylinder. A torsion spring restrains rotational movement of the inner cylinder. A dial scale is attached to the inner cylinder, and with rotation of the inner cylinder, indicates on a fixed pointer the angular displacement of the inner cylinder. The torque produced on the inner cylinder by rotation of the outer cylinder rotates the inner cylinder until the torque on the torsional spring is counter balancing the torque exerted by the fluid. At this point, a reading of the viscosity of the fluid may be taken.

However, direct-indicating rotational viscometers suffer from several problems. Primarily, a high degree of accuracy in shear stress readings is difficult to attain when reading from a scale. Fluctuations in the meter scale about an average position occur as a result of changing physical properties of the fluid and the presence of solid particles in the fluid as the outer cylinder is rotated. As a result, the operator reading the scale must interpolate the average position of the scale. The API recommendation Standard Procedure for Testing Drilling Fluids (APIRP 13B, 7th Edition, April 1978) suggest a that before reading the shear stress at a given shear rate, the dial reading should be allowed to come to a "steady value." Heretofore, rotational viscometers had to depend upon the operator's subjective determination of when a "steady value" has been attained. However, a slow drift in the steady dial reading may occur even with relatively stable readings on the dial, i.e. there are relatively small fluctuations in the dial's

position. This drift can be brought about by gradual change in the structure of the fluid. As a result, a quantitative determination of when a steady value or steady state condition of the fluid had been reached is difficult to attain in these prior-art-viscometers.

A further problem in direct-indicating rotational viscometers is the inability of the outer cylinder to change speeds quickly, and at the same time, be capable of maintaining accurate selected shear rate speeds. One such prior-art means for maintaining an accurate shear rate speed in a rotational viscometer is disclosed in U.S. Pat. No. 4,062,225. A phase locked loop (PLL) motor speed control circuit is shown for controlling the shear rate speed of the rotated outer cylinder. Attached to the motor shaft is a high inertia flywheel which functions to dampen out speed variations of the motor at the selected shear rates. Unfortunately, the inertia of the flywheel does not permit a rapid change in the motor speed as the shear rate is changed.

This inability to rapidly change speed becomes especially significant when the viscometer is measuring the "Gel" strength of the fluid. As recommended at Page 6 of the API bulletin identified above, when measuring "Gel" strength, the fluid is mixed at high speed for 10 seconds, stopped for 10 seconds and then run at 3 RPM. The maximum reading is recorded and process of mixing, stopping and running at a low RPM is repeated. For this procedure, it is implied that following the mixing step, the outer cylinder is immediately stopped to allow the fluid to reform. This would not be possible with a motor speed control system utilizing a flywheel to attain high accuracy shear rate control. The inertia of the flywheel requires a significant amount of time for the rotating outer cylinder to come to a stop.

Accordingly, it would be advantageous to provide a rotating type viscometer in which the shear rate speeds can be controlled to a high degree of accuracy, but at the same time, permit rapid changes in the rotation of the outer cylinder. It would also be advantageous to provide a viscometer that could determine quantitatively the steady state condition of the shear stress at each selected shear rate to achieve a high degree of accuracy in shear rate measurements. It would also be advantageous to provide a viscometer that could automatically and accurately measure the shear stress of a fluid at each of a pre-selected number of shear rates to obtain data values which will permit the piece-wise linear approximation to the shear stress profile of the fluid, particularly in the region of actual shear rate conditions encountered in the annulus of a well bore.

### SUMMARY OF THE INVENTION

In accordance with this invention, a rotational concentric cylinder viscometer for measuring the steady state shear stress of a fluid at a given shear rate is provided. The fluid, whose viscosity is to be measured, is contained between a rotatable outer cylinder and a spring loaded rotatable inner cylinder such that rotation of the outer cylinder produces a torque acting through the fluid on the inner cylinder thereby causing the inner cylinder to rotate. The total amount of rotation of the inner cylinder is related to the shear stress of the fluid as measured at the shear rate.

A microprocessor unit is provided to input and output data signals that will select the various shear rates of rotation of the outer cylinder, and measure the resulting amount of rotation of the inner cylinder at each selected



shear rate. A system clock provides several system clock signals to control the various functions and timing of the viscometer. A shear rate controller responds to the system clock and the shear rate selecting data from the microprocessor to rotate the outer cylinder at the selected constant angular velocity. An angular position indicator comprising an absolute value shaft encoder outputs a digital Gray code that is used to indicate the angular position of the inner cylinder.

A function selector means comprising a manually actuated keyboard is connected to the microprocessor input data lines for inputting both the mode of operation of the microprocessor and for inputting the various pre-selected shear rates at which the outer cylinder is to be rotated. A display means is provided to display both a measured shear stress and the angular velocity of the outer cylinder at which the display shear stress was measured.

The shear rate controller that controls the rate of rotation of the outer cylinder includes a programmable frequency divider that responds to the shear rate selecting data from the microprocessor to produce a phase detector clock signal. The phase detector clock signal is produced by dividing the feedback frequency signal obtained from a speed encoder connected to the outer cylinder. The speed encoder is an optical incremental encoder mechanically connected to the outer cylinder for rotation therewith. A motor drive means responds to the phase detector clock signal to generate the excitation signal to the motor that rotates the outer cylinder. This motor drive means includes a phase detector that responds to the system clock and the phase detector clock signal to generate a phase error signal that indicates the phase difference between the phase detector clock signal and a reference signal. Also included is an amplifier that filters and amplifies the phase error signal to produce the motor excitation signal. The motor used to rotate the outer cylinder is a DC-motor of the ironless armature type.

In another aspect of the invention, a method for obtaining the steady state shear stress of a fluid at a selected shear rate using the microprocessor controlled rotating-cylinder viscometer is also disclosed. The method comprises the steps of changing the outer cylinder rotational speed to a selected shear rate, determining a current average value for the angular position of the inner cylinder from a predetermined number of position measurements taken at a predetermined sample rate. The difference between a current average value and the last obtained average value is then obtained. The result is then compared with a predetermined number of degrees to determine if the steady state condition of shear stress has been attained. If not, the process of obtaining a current average value is repeated until the difference is less than or equal to the predetermined number of degrees, at which point the current average value is taken as the steady state shear stress.

In another aspect of the invention, a method for obtaining the instant "GEL" and 10 minute "GEL" using the microprocessor controlled rotating cylinder viscometer is also disclosed. The method comprises the programmed steps of mixing, waiting the appropriate delay time, and starting the outer cylinder at 3 RPM during which the highest shear stress reading obtained in an interval of 20 seconds, corresponding to one complete revolution of the outer cylinder, is stored for later recall.

In yet a further aspect of the invention, a method for obtaining the shear stress profile (shear stress vs. shear rate) of a fluid using the microprocessor controlled rotating-cylinder viscometer is disclosed. The method for determining the steady state shear stress at a selected shear rate is repeated for each of the pre-selected shear rates until all readings have been obtained.

### BRIEF DESCRIPTION OF THE DRAWINGS

The novel features believed characteristic of this invention are set forth in the appended claims. The invention and advantages thereof may best be understood by reference to the following detailed description of the illustrative embodiments read in conjunction with the accompanying drawings which form a part of this specification, and in which corresponding numerals indicate corresponding parts.

In the Drawings:

FIG. 1 is a block diagram representation of the microprocessor controlled rotating-cylinder viscometer of the present invention;

FIG. 2 is a circuit diagram of the phase locked loop controller of the present invention that controls the rate of rotation of the outer cylinder;

FIG. 3 is a circuit diagram of various blocks illustrated in FIG. 1 including the inner cylinder position indicating means, the I/O function decoder and the keyboard entry to the microprocessor;

FIG. 4 is a tabular listing of the program memory allocation for microprocessor 10 of the present invention; and

FIGS. 5-16 are software flow diagrams for the various routines for the programmed microprocessor 10.

### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS OF THE PRESENT INVENTION

Referring to the figures and first to FIG. 1, a block diagram of the rotational concentric cylinder viscometer of the present invention is shown. A microprocessor 10 functions to control the operations of the rotating viscometer, and to measure the resulting angular rotation of the inner cylinder indicative of the shear stress of the fluid at a given shear rate. The various elements of the viscometer are connected to the microprocessor 10 by way of three data buses, I/O ports P1 (P10-P17) and P2 (P20-P27), and data bus BUS. In the preferred embodiment of the present invention, microprocessor 10 is manufactured and sold by Intel Corporation as its MCS 8749 Microcomputer.

A crystal controlled system clock 20 is provided to generate various frequencies used to control the functions and timing of the viscometer. In the preferred embodiment, system clock 20 outputs seven frequencies illustrated in the following Table 1:

TABLE 1

F	Frequency (Hz)
1	4,194,304
2	2,097,152
3	16,384
4	512
5	4
6	2
7	1

Frequency f1 is supplied to the microprocessor 20 for its internal timing, f2 is supplied to the programmable keyboard/display interface 72 (FIG. 3), f3 is supplied to the



keyboard encode chip 82 (FIG. 3), f4 is supplied to the motor speed control 28, f5 is supplied to the microprocessor 10 for digit display purposes, f6 is supplied to the motor speed indicator 24 and f7 is supplied to the elapsed timer contained in microprocessor 10.

Command control data is outputted by the microprocessor 10 to initiate various sequences in the elements of the viscometer or to cause data to be applied to the data buses. Operation of the microprocessor controlled viscometer is initiated from a keyboard assembly 12 having 18 buttons. Two of the 18 buttons are used to switch the power on and off to the circuits. The remaining 16 keys are labeled as follows: 10 of the keys contain the digits 0-9, one each of the remaining 6 keys are respectively labeled STORE, RECALL, ERASE, ENTER, AUTO, and GEL. The 10 keys marked with digits 0-9 are used to input numeric data to select the speed for the various shear rates, and to select the memory locations in conjunction with the "STORE" key. The "RECALL" key is used in conjunction with a series of numeric actuated keys to display a stored value. Any speed selected and inputted to the microprocessor 20 from the keyboard and not ENTERED can be erased with the "ERASE" without interfering with the previous selected speed. Automatic programs can be terminated by actuating the "ERASE" key. The functions performed by the "ERASE" key are as follows: It clears entries during a keyboard input sequence; it clears any "Err" messages; it causes an exit from the "GEL" routine while counting elapsed time; it causes an exit from the "GEL" routine during the 20-sec comparison interval; it causes an exit from the "AUTO" sequence during settling time after speed change; it causes an exit from "RECALL"; it causes an exit from "STORE" with no effect on memory contents; it causes an exit during the 10 second mixing interval; and it causes a jump to the calibration routine if depressed during power-up.

All selected speeds have to be entered with the "ENTER" key. The functions of the "ENTER" key are as follows: It loads the keyboard input data into the programmable divider 26; it starts the comparison routine in "GEL" at the displayed elapsed time; and it causes a return to the main program after completion of the calibration routine. All keyboard actuated numbers are displayed, with the keyboard indicator light on, until it is entered into memory. At that time, the display 16 will indicate the actual speed of the rotating outer cylinder and the resulting steady state shear stress. Any invalid number inputted will display an error "Err". The function of the "AUTO" and the "GEL" keys, will be described in detail below.

The viscometer of the present invention may be operated either in an automatic or a manual mode. In the automatic mode, "AUTO", the steady state shear stress for each of a predetermined number of preselected shear rates will be obtained and stored in memory. This data can then be used to obtain a piece-wise linear approximation to the viscosity profile of the fluid at various shear rates. In order to perform the automatic determination of the shear stress, the keyboard 12 key labeled "AUTO" must be depressed.

When the viscometer is operated in a manual mode, it is possible to preselect the speed at which the outer cylinder is to be rotated by inserting the desired rotation rate via the numbered keys and depressing the "ENTER" key. The outer cylinder will begin to rotate at the inputted speed. When the operator is satisfied that the

steady state value of the shear stress has been attained, depressing the "STORE" button and a numeric button corresponding to a memory location will cause the microprocessor 10 to read both the actual speed of the outer cylinder, via motor speed indicator 24, and the resulting shear stress. This data will be stored in the memory location corresponding to the numeric key depressed after the "STORE" key. During the manual mode of operation, the measured speed and shear stress data are outputted to two 3-digit displays 16.

Still referring to FIG. 1, control of the rotation of the outer cylinder 36 is achieved through the combination of microprocessor 10, programmable divide-by-N counter 26, motor speed control 28 and the DC-motor 30. In particular, the pre-selected or manually entered shear rate data is outputted by microprocessor 10 as a divide factor N to programmable divide counter 26. Counter 26 divides a feedback frequency signal 5 outputted by an optical incremental encoder 32 that is coupled to the rotation of motor 30. The rate of rotation of motor 30 controls the frequency of the feedback frequency signal 5 that is inputted both to the programmable divide counter 26 and to the motor speed indicator 24. The function of motor speed indicator 24 is to produce a digital code word that indicates the actual rotational speed of the motor that is occurring. The output of motor speed indicator 24 is applied as data input to the microprocessor 10, and is read under control of the micro processor.

The output of programmable divide-by-N counter 26 is applied to motor speed control 28 where it is compared with a reference frequency supplied by system clock 20. For the presently preferred embodiment the reference clock signal is f4, 512 Hz. The phase difference between the reference frequency and the output of the programmable counter 26 is a phase error signal that is filtered and amplified to produce the excitation signal to DC-motor 30. Rotation of the output shaft of motor 30 is reflected as a feedback frequency signal 5. When the feedback frequency signal 5 reaches the proper frequency, as determined by the divide factor N and the reference frequency from system clock 20, the excitation signal to DC-motor 30 will remain constant to cause motor 30 to run at the desired velocity. Variations of the motor 30 speed will be reflected as a variation in the feedback frequency signal 5. These variations will produce an appropriate change in the motor excitation signal to bring the motor speed back to the desired setting.

In the presently preferred embodiment of the invention, the DC-motor 30 is a permanent magnet DC-motor having an armature wound on a hollow non-magnetic core. Motors such as these having no magnetic material in the armature or rotor are known as "Ironless motors." A suitable motor of the ironless type is manufactured and sold by Interelectric Corporation of Switzerland as a Maxom DC-motor series 2332 and by Dr. Faulhaber Company of Germany as series 3557. An ironless motor is desirable because of its low inertia and fast acceleration. These features enable the motor to achieve a fast reacting precision motor control, and a short settling time.

Referring now to FIGS. 1 and 2, the concentric rotating cylinders 36 respond to rotation of the DC-motor 30 to produce a corresponding rotation in the shaft of the inner cylinder. This angular rotation is monitored by absolute value shaft encoder 38 that outputs a digital code word indicative of the actual angular position of



the inner cylinder. This digital data is inputted to a code converter 34 whose output is coupled onto one of the microprocessor 10 data buses, P10-P17. In the presently preferred embodiment, absolute value shaft encoder 38 outputs a 9-bit Gray code where only one bit line is permitted to change per interval of angular rotation of the encoder shaft. The position encoder 38 is graduated from -9° to +350° deflection. The code converter 34 converts the Gray code of encoder 38 into its BCD equivalent values. Code converter 34 consists of a ROM having the BCD equivalent codes stored in memory locations that are addressed by the output code words from the encoder 38. Each output code word from encoder 30 serves as a unique address to the code converting ROM.

To perform the code conversion, a 9-bit address to the code converting ROM contained in converter 34 is generated by the encoder 38. The most significant bit of the 10-bit address to the ROM is inputted from the microprocessor 10 on the P23 data bus line. Under program control, microprocessor 10 causes I/O decoder 80 to generate the strobe D to multiplex onto data bus lines P10-P17 two BCD digits of the three BCD digits that represent the angle of the inner cylinder's shaft. Data bus P23 is then set and strobe D generated to output from the code converting ROM the third BCD digit along with the + or - sign for the angle. The encoder disk 35 of encoder 38 is properly etched to produce the Gray code. To produce the electrical signals to read the etched disk 35, the present invention uses a Texas Instruments light emitting diode array T1L 49 and a photosensitive transistor array T1L629.

Turning now to FIG. 2, the motor speed indicator 24, programmable divider counters 26 and motor speed control 28 are illustrated in more detail. The encoder 32 consists of a rotatable disc 40 having slits or means therein for permitting the passage of light from light emitting diode D1 to reach the base of light sensitive transistor Q1. The output of transistor Q1 is amplified and applied to comparator A1 to produce a digital signal at the input of exclusive OR gates 44. The output of exclusive OR gates 44 comprises the feedback frequency signal 5. For the preferred embodiment, encoder 32 produces 3840 pulses per revolution on the output shaft of outer cylinder 36. At a rotor speed of 600 RPM, encoder 32 produces a frequency of 76,800 pulses per second.

As previously discussed, the feedback frequency signal 5 is applied to programmable divider 26 to produce the phase frequency signal on its output. This signal is applied as one input to two input NAND gate 60. Inverter 58 responds to a control signal from data bus P25 to provide the other input to NAND gate 60. Control signal P25 functions to select the phase frequency clock from programmable divider 26 as the input signal to phase detector 68 for all rotor speeds above 1 RPM. When a rotor speeds of 1 RPM is desired, the microprocessor 10 outputs a control signal on P25 to select the feedback frequency signal 5 as the frequency to be applied directly to phase detector 68 by way of NAND gates 62 and 64.

As illustrated in FIG. 2, NAND gates 58, 60, 62 and 64 function as an AND/OR gate to select either the output of the programmable divider 26 or the feedback frequency signal 5 as the input clock signal to phase detector 68 depending upon the state of data bus line P25. The divide parameter N from the divide-by-N counter 26 is supplied to the programmable divider 26

from the microprocessor 10 BUS. Programmable divider 26 is manufactured and sold by Intel Corporation as a model 8253 divide-by-N counter. Prescaler counter 52 which forms part of the motor speed indicator 24 is contained in the Intel 8253 chip.

Also inputted to the motor speed control circuit 28 is a second control signal, P27, from microprocessor 10. This signal enables the phase detector 68 to generate an excitation signal to the motor 30 to cause it to begin rotation. When data bus line P27 is not active, motor 30 will not rotate. When enabled by P27, the 512 Hz frequency signal f4 from the system clock 20 is applied to phase detector 68 which, for the preferred embodiment of the present invention, is manufactured and sold by Motorola Incorporated as a phase/frequency detector MC14568. This particular phase detector contains a divide by 4 internal counter to reduce the 512 Hz reference signal down to 128 Hz. The phase detector 68 produces an error signal representative of the phase difference between the signal from the encoder 32 divided by the division ratio N, and the 128 Hz reference signal. This error signal is filtered and amplified by amplifier A2 and power amplifier Q3 to provide the excitation signal to the motor. It should be clear that the values of the various circuit components, voltages, and their manufacturer type numbers depicted in FIG. 2 and described above will vary depending upon the intended use. In a presently preferred embodiment use in connection with a phase locked loop control system to control the speed of rotation of motor 30, Table 2 below sets out exemplary values which have been found satisfactory.

TABLE 2

REFERENCE	VALUE (ohms)	REFERENCE	TYPE or VALUE
R1	120	C1	820pf
R2	1K,VAR	C2,C6	.1 f
R3	47K	C3,C7	.001 f
R4,R6,R12	100K	C4	1 f
R5,R9	1K	C5	10 f
		D1	OPTRON INC. 81355
R7, R13	100K,VAR	D2	55T
R8	330K	Q1	OPTRON INC. 81355
R10, R11	10K	Q2	2N4123
R14	6.8K	Q3	MJE6044, 2N6388
R15	39	A1	LM311, MLM311 (National)
		A2	LM358, MLM358 (Motorola)

Still referring to FIG. 2, the motor speed indicator 24 for determining the actual speed of rotation of the motor 30 is shown. The feedback frequency signal 5 is applied to a divide by 64 pre-scaler 52 to reduce the frequency generated by encoder 32. This reduced frequency is applied to a 3-digit BCD counter 54 that is periodically cleared by the 2 Hz reference signal f6 from the system clock 20. Counter 54 is manufactured and sold by Motorola as model MC14553. The 3-digit counter 54 functions to generate a 3-digit BCD code equal to the number of pulses produced in a time interval of one second, and multiplexes each digit sequentially onto its output. The multiplexed output of the 3-digit counter 54 is applied to a tri-state buffer circuit 56 that responds to function strobe A from decoder 22 to apply the BCD digits onto the microprocessor bus P10-P16. Also strobed onto the data bus lines P10-P16 are the digit selection bits that indicate which of the three BCD digits from counter 54 is currently being



multiplexed into the buffer 56. For the presently preferred embodiment, buffer 56 is a Texas Instruments tri-state buffer model SN74LS244. The control signal A is generated in response to function control data outputted by microprocessor 10 when a reading of the speed of rotation of the outer cylinder 36 is desired.

Turning now to FIG. 3 in which various functions illustrated in FIG. 1 are illustrated in more particular detail, the absolute value shaft encoder 38 is shown responding to the rotatable inner cylinder 37 of the concentric rotating cylinders. The encoder 38 functions in a similar manner to the optical encoder 32 in that light emitting diodes D3-D11 are used to produce light that is applied to the bases of transistors Q4-Q12 through a disc that rotates with the shaft of the inner cylinder 37. Each output of transistors Q4-Q12 is applied to a respective buffer gate 70 to produce the 9-bit digital code representative of the angular position of the inner cylinder 37. As previously mentioned, the 9-bit Gray code word is applied to code converter 34 for conversion to BCD digits that are applied to the microprocessor data bus lines P10-P17. The data from the encoder 38 is applied to the bus through the code converter 34 in response to function strobe D outputted by the I/O function decoder 22.

The I/O function decoder chip 22 which generates the various function strobe signals is of a 1-of-8 decoder that responds to the microprocessor 10 data bus lines P20-P22 to produce eight strobes used to control various sequences and to enable data to be inputted to the microprocessor 10 data buses. For the preferred embodiment, decoder 22 is manufactured and sold by Intel Corporation as model 8205. Also illustrated in FIG. 3 is the system clock 20. It will be appreciated by those of ordinary skill in the art that the basic reference frequency produced by a crystal controlled oscillator may be divided down by counters to produce the reference frequencies illustrated. Accordingly, the circuits for this function are not shown and discussed herein.

Further illustrated in FIG. 3 is the keyboard 12 consisting of 16 keys that are used to input the data to the microprocessor 10 needed to operate the viscometer. The output from the various keys are applied to the keyboard encoder 82. Encoder 82 responds to the 16 KHz reference frequency f3 from the system clock 20 to produce the 4-bit binary code eventually applied through the buffer gates 84 to the microprocessor 10 input data bus lines P10-P13. Function strobe B enables gate 84 to apply the 4-bit code words onto the data lines. Also produced in the keyboard encoder 82 is a strobe signal that is applied to the display interface 72 which generates an interrupt request signal to microprocessor 10. The strobe signal is only produced when one of the numeric keys is depressed. The actuation of one of the mode control keys to the microprocessor 10 does not produce an immediate response from the microprocessor. Rather, the internal software routines will periodically read the output from the keyboard buffer gates 84 and determine if any of the function keys have been depressed.

Still referring to FIG. 3, the two 3-digit displays 16 are shown diagrammatically as composed of 7-segment display digits 74 that respond to digital driver 76 and display interface 72. The digits to be displayed in the digits 74 are loaded from the microprocessor 10 BUS lines into the display interface 72 on the occurrence of the control signal E from the I/O function decoder 22. The various digits to be displayed are serially loaded

into display interface 72 which time multiplexes the digits onto the control lines to the digits of display 74. In this manner, the data stored in display interface 72 appears to be continuously displayed by the display unit 74. The presently preferred embodiment has used an Intel Corporation 8279 display chip for display 16.

As previously mentioned, the present invention is able to determine the steady state value for the shear stress at each of the preselected rotor speeds. Shortly after application of power to the electronic circuits, the microprocessor 10 measures the angular position of the inner cylinder at zero speed to determine if an offset error is present. This offset error is used to correct all further angular position measurements. Upon entry into the automatic mode by microprocessor 10, the outer cylinder is rotated at 700 RPM for a period of 10 seconds to mix the fluid. Mixing is required to cause a breakdown of all particle bonds for particles suspended in the fluid. At the end of the 10 second interval, the outer cylinder is then rotated at 600 RPM a period of 2 seconds to allow for the speed of the outer cylinder 36 to stabilize and become phase locked to the fixed reference signal from the system clock 20. Readings of the angular deflection as reflected from absolute value shaft encoder 38 are then taken at a sample rate of 128 samples/second for one second.

Characteristic of all concentric rotating cylinder viscometers is the presence of fluctuations in the angular position of the inner cylinder 37 due to random variations in the torque transmitted by the fluid to the inner cylinder 37. The magnitude and rate of the fluctuations may vary from fluid sample to fluid sample depending on the amount and size of the solid particles present. For the present invention, it is important that the sample rate of angular position readings of the inner cylinder 37 be at least two times the maximum rate of fluctuations present in the angular position of the inner cylinder. As the readings are taken, they are consecutively totalized until all of the readings have been taken. Then, the total is divided by the number of samples taken, 128, to give an average angular deflection of the inner cylinder position for that one second measuring interval. The presence of solid particles in the fluid being tested can exert intermittent or irregular forces on the cylinder thus causing the cylinder to oscillate about a mean angular position. The averaging sequence eliminates the need for a mental averaging by the operator to obtain this mean, and thereby reduces the possibility of an error in the reading.

Many fluids exhibit thixotropic properties and require a certain time period for particle bonds to reform after a reduction in shear rate. After all bonds that can exist at a particular shear rate have reformed, the shear stress will reach an equilibrium value and a reading may be taken. For this reason, the averaging sequence is performed again after a two second time interval. The next average value thus obtained is compared to the previous average value, and if the two values are within plus or minus one degree ( $\pm 1^\circ$ ) of each other, the current average value is accepted as the true equilibrium or steady state reading. If the current average value differs in magnitude by more than one degree from the previous average value, another two second time interval elapses and the averaging process is again repeated. This procedure continues until the difference between the present and previous readings are within the desired range.

After the steady state reading has been determined, that reading, along with the shear rate at which it was



obtained, is stored in a memory location for later recall and display. In one embodiment of the present invention, the rotational speeds of 600, 300, 200, 100, 60, 30, 15, 6 and 3 RPMs are recommended to obtain shear stress readings that will permit the piece-wise linear approximation of the shear stress versus shear rate curve, and allow the operator to make calculations on certain rheometric properties of the fluid. The various shear rates to obtain the shear stress profile would have been inputted into the microprocessor 10 prior to initiating the automatic mode of operation. The above described procedure for obtaining the steady state shear stress at a shear rate would then be repeated for each of the preselected and inputted shear rates until the measurements of shear stress at each of the speeds has been obtained.

In the automatic mode, after all of the preprogrammed shear rate readings have been measured and recorded, the microprocessor 10 proceeds to the GEL mode. This GEL mode is also separately selectable by the operator by depressing the button on the keyboard marked "GEL." Upon initial entry into the GEL mode, the outer cylinder 36 is rotated at 700 RPM for a period of 10 seconds. Again to mix the fluid under test and to break down particle bonds that give the fluid thixotropic properties. After the mixing period, the outer cylinder rotation is stopped and the elapsed time counter interval to microprocessor 10 is initiated to generate the elapsed time. The microprocessor 10 internal elapse time counter is incremented once every second by f7, and upon reaching 10 seconds, the outer cylinder 36 is

rotated at 3 RPM. Immediately as the outer cylinder 36 starts to rotate at 3 RPM, readings of the angular displacement of the inner cylinder 37 are taken in rapid succession for a period of 20 seconds corresponding to one complete revolution of the outer cylinder 36. During this 20 second time period, the largest value is retained, and after the 20 second period, this maximum value is stored in an operator specified addressable memory location for later recall. This stored reading corresponds to the initial GEL strength of the fluid under test. The fluid is then mixed as before and then the same sequence of measurements occur after a second time period of 10 minutes. The elapse timer can be interrupted manually by depressing the ENTER button and with this manual override, any time GEL, up to 600 seconds can be taken. When all measurements are completed, the microprocessor 10 exits the GEL routine and leaves the outer cylinder rotating at 3 RPM with the display showing the speed and corresponding shear stress reading.

Referring now to FIGS. 5-16, software flow diagrams for various functions performed by the present invention are shown. These flow diagrams illustrate the functions performed by microprocessor 10 in response to the various modes of operation keyed from keyboard 12. The following is an assembler language listing of the microprocessor 10 program that implements the flow diagrams of FIGS. 5-16. It will be appreciated by those skilled in the art that other routines other than those illustrated and described herein could be programmed into microprocessor 10 and achieve the same results.

```

1  TITLE( 8749 2K RHEOMETER PROGRAM V2
2  MACROFILE  DEBUG XREF PAGEWIDTH (80) NOGEN
3  WRITE      MACRO PORT2,DBUS
4              MOV A, #PORT2
5              OUTL P2,A
6              MOV A, #DBUS
7              OUTL BUS,A
8              ENDM
9  PREP      MACRO AND,OR,PORT
10             ANL P2,#AND
11             ORL P2,#OR
12             MOV A,#PORT
13             OUTL BUS,A
14             ENDM
15  TAB7      EQU 0ECH
16             MOV A,#0GH           ;RESET 8279
17             OUTL P2,A
18             NOP
19             ANL P2,#00H
20             MOV RO,#18H         ;CLEAR RAM 18 to 7F
21             MOV R7,#68H
22             CLR A
23  CLEAR:    MOV @RO,A
24             INC RO
25             DJNZ R7,CLEAR
26             WRITE 15H, 06H     ;SET KYDB/DISP MODE
31             MOV A,#35H
32             OUTL BUS,A         ;PROGRAM 8279 CLOCK
33             MOV A,#0C3H
34             OUTL BUS,A         ;CLEAR ALL IN 8279
35             MOVA,#80H
36             MOV T,A
37             STRT T
38             JTF +4
39             JMP -2
40             STOP TCNT
41             WRITE 1BH, 16H     ;PROGRAM 8253 CNT 0
46             MOV A,#56H
47             OUTL BUS,A         ;PROGRAM 8253 CNT 1
48             MOV A,#0B5H
49             OUTL BUS,A         ;PROGRAM 8253 CNT 2
50             WRITE 0BH,40H     ;LOAD CNT 1
55             WRITE 13H,00H
60             MOV A,#01H

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61      OUTL BUS,A
62      MOV A,#02H
63      OUTL P2,A          ;SET FOR KYBD INPUT
64      JMP OFFADJ
65  KYBD:  ANL P2,#0A0H
66         ORL P2,#02H
67         JNI FUNC
68         MOV R7,#04H
69  STAT:  JNI DATIN
70         MOV R0,#18H
71         ANL P2,#0E0H
72         ORL P2,#42H
73         IN A,P1          ;INPUT KYBD DATA
74         ANL A,#0FH
75         XCH A,@RO
76         INC R0
77         XCH A,@RO
78         INC R0
79         MOV @RO,A
80         MOV R0,#18H
81         MOV R1,#1BH
82         MOV R6,#03H
83         MOV A,@RO
84         MOV @R1,A
85         INC R0
86         INC R1
87         DJNZ R6,$-4
88         CALL DISP
89         CALL CLRIF
90         DJNZ R7,STAT
91         JMP ERROR
92  DATIN: ANL P2,#0E0H
93         ORL P2,#02H
94         IN A,P1
95         ANL A,#0FH
96         MOV R5,A
97         XRL A,#0AH
98         JZ ERASE          ;TEST FOR 'ERASE'
99         MOV A,R5
100        XRL A,#0BH
101        JZ ENTIN          ;TEST FOR 'ENTER'
102        JMP STAT
103  ERASE: CLR A
104         MOV R0,#18H
105         MOV R7,#06H
106         MOV @RO,A
107         INC R0
108         DJNZ R7,$-2
109         JMP KEYOUT
110  ERROR: MOV R0,#1BH          ;LOAD 'ERR' MESSAGE
111         IMP (13H, 13H)
112         MOV @RO, #M
113         INC R0
114         ENDM
119        MOV @RO,#0FH
120        CALL DISP
121        ANL P2,#0E0H
122        ORL P2,#02H
123        IN A,P1
124        ANL A,#0FH
125        XRL A,#0AH
126        JZ ERASE          ;TEST FOR 'ERASE'
127        JMP $-7
128  ENTIN: MOV R0,#1AH          ;TOTAL KYDB REGISTER
129         MOV A,@RO
130         DEC R0
131         ADD A,@R0
132         MOV R7,A
133         DEC R0
134         ADD A,@R0
135         JZ MOTOFF
136         DEC A
137         JNZ GOENT
138         MOV A,R7
139         JNZ GOENT
140         MOV a,#0A0H
141         OUTL P2,A
142         MOV R0,#18H
143         CLR A
144         MOV @R0,A
145         INC R0
146         MOV @R0,A

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-continued

147		INC R0	
148		MOV @R0,A	
149		JMP KEYOUT	
150	MOTOFF:	CLR A	;TURN MOTOR OFF
151		OUTL P2,A	
152		JMP KEYOUT	
153	GOENT:	CALL ENTER	
154		JMP KEYOUT	
155	FUNC:	IN A,P1	;CHECK KYBD FUNCTIONS
156		ANL A,#0FH	
157		MOV R5,A	
158		XRL A,#0CH	
159		JNZ \$+4	
160		JMP GEL	
161		MOV A,R5	
162		XRL A,#0DH	
163		JNZ \$+4	
164		JMP AUTO	
165		MOV A,R5	
166		XRL,#0EH	
167		JNZ \$+4	
168		JMP RECALL	
169		MOV A,R5	
170		XRL A,#0FH	
171		JNZ \$+4	
172		JMP STORE	
173		JMP KEYOUT	
174	GEL:	CALL GMESS	
175		CALL MIX	
176		MOV R4,#0F6H	;R4=CNT START CODE
177		MOV R5,#26H	;R5=STORAGE ADDRESS M
178		MOV R6,#01H	;R6=NO. OF TF OVERFLO
179		MOV R7,#00H	;R7=100's of SEC CNTR
180	STOPMO:	CLR A	
181		OUTL P2,A	
182		MOV A,#0FEH	
183		MOV T,A	
184		STRT CNT	
185		MOV A,R5	
186		MOV R0,A	
187		CLR A	
188		MOV @R0,A	;CLEAR STORAGE MEMORY
189		DEC R0	
190		MOV @R0,A	
191		DEC R0	
192		MOV @R0,A	
193		JTF \$+4	
194		JMP \$-2	
195		STOP TCNT	
196		CALL GMESS	
197	COUNT:	MOV A,R4	
198		MOV T,A	
199		STRT CNT	;START TIMER
200	TFTEST:	JTF LOOPS	
201		MOV A,#02H	
202		OUTL P2,A	
203		IN A,P1	
204		ANL A,#0FH	
205		MOV R3,A	
206		XRL A,#0BH	;TEST FOR 'ENTER'
207		JNZ \$+4	
208		JMP GOGEL	
209		MOV A,R3	
210		XRL A,#0AH	;TEST FOR 'ERASE'
211		JZ GELOUT	
212		MOV A,T	
213		CPL A	
214		ADD A,R4	
215		CPL A	
216		MOV R2,A	
217		MOV R1,#00H	
218		MOV R0,#1BH	
219	TENS:	MOV A,R2	
220		CPL A	;CONVERT TIMER VALUE
221		ADD A,#0AH	;TO 2 BCD NUMBERS
222		JC ONES	
223		INC R1	
224		CPL A	
225		MOV R2,A	
226		JPM TENS	
227	ONES:	MOV A,R2	
228		MOV @R0,A	

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229      INC R0
230      MOV A,R1
231      MOV @R0,A
232      INC TO
233      MOV A,R7
234      MOV @R0,A
235      CALL DISP
236      JMP TFTEST
237  LOOPS:  DEC R6
238          MOV A,R6
239          JZ GOGEL
240          INC R7
241          JMP COUNT
242  GOGEL:  STOP TCNT
243          MOV A,#0ECH
244          MOV T,A
245          STRT CNT
246          MOV A,R5
247          ADD A,#1FH
248          MOV R0,A
249          MOV R1,A
250          MOV @R0,#0EH      ;LOAD 'G-N' MESSAGE
251          DEC R0
252          MOV @R0,#0AH
253          DEC R0
254          MOV A,R1
255          JBO $+6
256          MOV @R0,#02H
257          JMP $+4
258          MOV @R0,#01H
259          MOV R0,#18H
260          MOV R3,#02H
261          MOV @R0,#03H      ;LOAD 003 RPM
262          INC R0
263          CLR A
264          MOV @R0,A
265          INC R0
266          MOV @R0,A
267          MOV R0,#1BH
268          DJNZ R3,$-9
269          CALL ENTER
270          CALL DISP
271  MAGCOM: CALL STRESS
272          MOV R0,#20H      ;PREVENT NEG READINGS
273          MOV A,@R0
274          XRL A,#0AH
275          JZ MAGOUT
276          MOV A,R5
277          MOV R1,A
278          MOV A,@R1
279          MOV R3,A          ;R3=OLD DATA
280          MOV R0,#20H
281          MOV A,@R0
282          MOV R2,A          ;R2=NEW DATA
283          CALL COMP
284          JF1 MAGOUT        ;F1=[R3 R2]
285          JNZ STOMAX        ;[A=0] = [R3=R2]
286          CLR C
287          DEC R0
288          DEC R1
289          MOV A,@R1
290          DEC R1
291          SWAP A
292          ADD A,@R1
293          MOV R3,A
294          MOV A,@R0
295          DEC R0
296          SWAP A
297          ADD A,@R0
298          MOV R2,A
299          CALL COMP
300          JF1 MAGOUT
301          JNZ STOMAX
302          JMP MAGOUT
303  STOMAX: CALL STODAT      ;STORE NEW DATA
304  MAGOUT: JTF AGAIN
305          MOV A,#82H
306          OUTL P2,A
307          IN A,#82H
308          ANL A,#0FH
309          XRL A,#0AH      ;TEST FOR 'ERASE'
310          JNZ MAGCOM

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311      JMP GELOUT
312  AGAIN:  MOV A,R7
313          JNZ GELOUT
314          CALL GMESS
315          CALL MIX
316          MOV R4,#9CH      ;RESET PARAMETERS
317          MOV R5,#23H
318          MOV R6,#06H
319          JMP STOPMO
320  GELOUT:  STOP TCNT
321          CLR F1
322          JMP KEYOUT
323  AUTO:    CALL AUMESS
324          CALL MIX
325          MOV R4,#08H
326          MOV R6,#0D4H      ;START OF SPEED DATA
327  LOAD:    MOV A,#0FEH
328          MOV T,A          ;START 2 SEC TIMER
329          STRT CNT
330          MOV R0,#1AH
331          MOV R1,#1DH
332          MOV R7,#03H
333          MOV A,R6
334          MOVP3 A,@A      ;LOAD SPEED DATA
335          MOV @R0,A
336          MOV @R1,A
337          DEC R0
338          DEC R1
339          INC R6
340          DJNZ R7,$-7
341          MOV A,R4
342          INC A
343          RL A
344          ADD A,R4
345          ADD A,#41H
346          MOV R1,A
347          MOV R0,#1BH
348          MOV R7,#03H
349          MOV A,@R0
350          MOV @R1,A
351          INC R0
352          INC R1
353          DJNZ R7,$-4
354          CALL ENTER
355          CALL AUMESS
356          CALL DISP
357          MOV A,#82H
358          OUTL P2,A
359          CLR A
360          MOV R7,#04H      ;CLEAR RAM 7AH TO 7DH
361          MOV R0,#7AH
362          MOV @R0,A
363          INC R0
364          DJNZ R7,$-2
365  RUN:     JTF STEP
366          IN A,P1          ;TEST FOR ERASE
367          ANL A,#0FH
368          XRL A,#0AH
369          JNZ RUN
370          JMP KEYOUT
371  STEP:    STOP TCNT
372          MOV R7,#80H
373          SEL RB1
374          CLR A
375          MOV R4,A
376          MOV R5,A
377          MOV R0,#20H
378          SEL RB0
379  BCDHEX:  CALL STRESS
380          MOV R0,#20H      ;PREVENT NEG READINGS
381          MOV A,@R0
382          XRL A,@9AH
383          JNZ $+8
384          CLR A
385          MOV @R0,A
386          DEC R0
387          MOV @R0,A
388          DEC R0
389          MOV @R0,A
390          SEL RB1
391          MOV A,#0BBH
392          MOV T,A

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393	STRT T	
394	JTF \$+4	
395	JMP \$-2	
396	STOP TCNT	
397	CLR C	
398	MOV A,@R0	
399	MOV R2,A	
400	RL A	
401	RL A	
402	MOV R3,A	
403	RL A	
404	RL A	
405	RL A	
406	MOV R2,A	
407	ADD A,R3	
408	MOV R3,A	
409	MOV A,R2	
410	RL A	
411	ADD A,R3	
412	MOV @R0,A	
413	JNC \$+4	
414	INC R5	
415	CLR C	
416	MOV A,R7	
417	RL A	
418	MOV R3,A	
419	RL A	
420	RL A	
421	ADD A,R3	
422	MOV R7,A	
423	ADD A,R6	
424	ADD A,@R0	
425	JNC \$+4	
426	INC R5	
427	CLR C	
428	ADD A,R4	
429	JNC \$+4	
430	INC R5	
431	CLR C	
432	MOV R4,A	;R4 CONTAINS HEX SUM
433	SEL RB0	;R5 CONTAINS OVERFLOW
434	DJNZ R7,\$+4	
435	JMP \$+4	
436	JMP BCDHEX	
437	SEL RB1	;CALC AVERAGE VALUE
438	CLR F0	
439	CLR C	
440	MOV A,R5	
441	RLC A	
442	MOV R5,A	
443	MOV A,R4	
444	JB7 \$+4	
445	JMP \$+5	
446	MOV A,R5	
447	INC A	
448	MOV R5,A	
449	MOV R0,#7CH	;DETERMINE IF SYSTEM
450	MOV R1,#7BH	;WAS IN EQUILIBRIUM
451	JNC \$+6	
452	MOV @R1,#01H	
453	JMP \$+4	
454	MOV @R1,#00H	
455	MOV A,@R0	
456	XRL A,R5	
457	JZ EQCARR	
458	DEC R1	
459	MOV A,R5	
460	ADD A,#01H	
461	JNC \$+6	
462	MOV @R1,#01H	
463	JMP \$+4	
464	MOV @R1,#00H	
465	XRL A,@R0	
466	JZ EQCARR	
467	MOV A,R5	
468	CPL A	
469	ADD A,#01H	
470	CPL A	
471	MOV R7,A	
472	JNC \$+6	
473	MOV @R1,#00H	
474	JMP \$+6	



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475      INC R1
476      MOV A,@R1
477      DEC R1
478      MOV @R1,A
479      MOV A,R7
480      XRL A,@R0
481      JZ EQCARR
482 UNSTAB:  MOV A,R5
483          MOV @R0,A
484          MOV R1,#7BH
485          INC R0
486          MOV A,#R1
487          MOV @R0,A
488          MOV A,#0FEH
489          MOV T,A
490          STRT CNT
491          SEL RB0
492          JMP RUN
493 EQCARR:  MOV R0,#7DH
494          MOV R1,#7AH
495          MOV A,@R0
496          XRL A,@R1
497          DEC R0
498          JNZ UNSTAB
499          MOV A,@R1      ;DATA ACCEPTED
500          CPL A
501          JB0 $+4
502          CPL F0
503          CLR C
504          MOV R0,#20H      ;CONVERT HEX TO BCD
505          MOV $7,#00H
506          MOV @R0,#00H
507 NSD:    MOV A,R5
508          CPL A
509          ADD A,#0AH
510          JC LSD
511          INC R7
512          CPL A
513          MOV R5,A
514          JMP NSD
515 LSD:    MOV A,R5
516          MOV R6,A
517 HUNS:   MOV A,R7
518          CPL A
519          ADD A,#0AH
520          JC ADD256
521          INC@R0
522          CPL A
523          MOV R7,A
524          JMP HUNS
525 ADD256: JF0 $+4
526          JMP AVOUT
527          MOV A,R6
528          ADD A,#06H
529          MOV R6,A
530          MOVA,R7
531          ADD A,#05H
532          MOVR7,A
533          MOV A,@R0
534          ADD A,#02H
535          MOV @R0,A
536          MOV A,R6
537          ADD A,#0F6H
538          JNC $+5
539          INC R7
540          MOV R6,A
541          CLR C
542          MOV A,R7
543          ADD A,#0F6H
544          JNC $+5
545          MOV R7,A
546          INC @R0
547          CLR C
548 AVOUT:  SEL RB0
549          MOV A,R4
550          INC A
551          RL A
552          ADD A,R4
553          ADD A,#24H
554          MOV R1,A      ;STORE STRESS READING
555          MOV R0,#20H
556          MOV R7,#03H

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557      MOV A,@R0
558      MOV @R1,A
559      DEC R0
560      DEC R1
561      DJNZ R7,$-4
562      DJNZ R4,$+4
563      JMP GEL
564      JMP LOAD
565  RECALL:  ANL P2,#0A0H
566          ORL P2,#02H
567          MOV A,#0FBH
568          MOV T,A
569          STRT CNT          ;START 5 SEC. TIMER
570          JTF $+9
571          IN A,P1
572          ANL A,#0FH
573          XRL A,#0AH
574          JNZ $+4
575          JMP KEYOUT
576          JN1 $-11
577          IN A,P1          ;READ MEMORY LOCATION
578          ANL A,#0FH
579          MOV R4,A
580          RL A
581          ADD A,R4
582          MOV R4,A
583          ADD A,#21H
584          MOV R1,A
585          MOV R0,#1EH      ;LOAD STRESS DATA
586          MOV R7,#03H
587          MOV A,@R1
588          MOV @R0,A
589          INC R0
590          INC R1
591          DJNZ R7,$-4
592          MOV A,R4          ;CALCULATE RPM ADDRESS
593          ADD A,#40H
594          MOV R1,A
595          MOV R0,190 1BH
596          MOV R7,#03H
597          MOV A,@R1
598          MOV @R0,A
599          INC R0
600          INC R1
601          DJNZ R7,$-4
602          CALL DISP
603          CALL CLRIFIF
604          JMP RECALL
605  STORE:  IN A,P1
606          ANL A,#0FH
607          MOV R5,A
608          XRL A,#0AH
609          JNZ $+4
610          JMP KEYOUT
611          JN1 STORE
612          MOV A,R5          ;CALC STRESS ADDRESS
613          RL A
614          ADD A,R5
615          MOV R4,A
616          ADD A,#23H
617          MOV R5,A          ;CALCULATE RPM ADDRESS
618          MOV A,R4
619          ADD A,#40H
620          MOV R4,A
621          CALL STODATE      ;STORE STRESS DATA
622          MOV A,R4
623          MOV R1,A
624          MOV R0,#1BH      ;STORE RPM DATA
625          MOV R3,#03H
626          MOV A,@R0
627          MOV @R1,A
628          INC R0
629          INC R1
630          DJNZ R3,$-4
631          CALL CLRIFIF      ;CLEAR FIFO OF 8279
632          JMP KEYOUT
633  KEYOUT: STOP TCNT
634          JTO $+4
635          JMP $+13
636          JF1 $+9
637          CALL RPM
638          CALL STRESS

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639      CALL DISP
640      CPL F1
641      JMP KYBD
642      CLR F1
643      JMP KYBD
644      ORG 400H
645  OFFADJ:  MOV R0,#7FH
646          CLR A
647          MOV R3,A
648          MOV @R0,A
649          MOV A,#0FEH
650          MOV T,A
651          STRT CNT          ;START 2 SEC. TIMER
652          JTF $+4
653          JMP $-2
654          STOP TCNT
655  CAL:    ANL P2,#0A0H
656          ORL P2,#02H
657          IN A,P1
658          ANL A,#0FH
659          XRL A,@0AH
660          JNZ ADJUST
661          CALL STRESS
662          CALL DISP
663          JMP CAL
664  ADJUST:  MOV A,#0FEH
665          MOV T,A
666          STRT CNT          ;START 2 SCE. TIMER
667          JIF $+4
668          JMP $-2
669          STOP TCNT
670          ANL P2,#0A0H
671          ORL P2,#04H
672          IN A,P1          ;READ OFFSET
673          MOV R2,A
674          ORL P2,#08H
675          IN A,P1
676          ANL A,#0FH
677          MOV R0,@7EH
678          MOV @R0,A
679          CALL COMP
680          JZ $+6
681          MOV A,R2
682          MOV R3,A
683          JMP ADJUST
684          MOV R1,#7FH
685          MOV A,@R0
686          JNZ $+6
687          MOV A,R2
688          MOV @R1,A
689          JMP $+6
690          MOV A,R2
691          ADD A,#80H
692          MOV @R1,A
693          CLR C
694          CLR F1
695          JMP KYBD
696  DISP:   PREP 0F0H, 15H,90H
697          ANL P2,#0EFH          ;SET C/D LOW
698          MOV R0,#1BH
699          MOV R2,#06H
700  EACH:   MOV A,@P0
701          ADD A,#TAB7
702          MOVP3 A,@A
703          OUTL BUS,A
704          INC R0
705          DJNZ R2,$+3
706          RETR
707          JMP EACH
708  STRESS: ANL P2,#0A0H
709          ORL P2,#04H
710          MOV R0,#20H
711          IN A,P1          ;READ LSD AND NSD
712          MOV R1,A
713          ORL P2,#08H          ;SET A9 HIGH
714          NOP
715          IN A,P1          ;READ MSD
716          ANL A,#0FH
717          MOV @R0,A
718          DEC R0
719          MOV A,R1
720          SWAP A
721
722
723
724

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725		ANL A,#0FH	
726		MOV @R0,A	;LOAD DATA TO REGISTER
727		DEC R0	
728		MOV A,R1	
729		ANL A,#0FH	
730		MOV @R0,A	
731		CLR C	
732		MOV R0,#20H	
733		MOV \$1,#03H	
734		MOV A,@R0	
735		ADD A,@0F5H	
736		JNC 4+4	
737		JMP STRESS	
738		DEC R0	
739		DJNZ R1,\$-8	
740		MOV R0,#7EH	
741		MOV R1,#7FH	
742		MOV A,@R1	
743		MOV @R0,A	
744	NEWOFF:	MOV R0,#7EH	
745		MOV A,@R0	
746		MOV R1,A	
747		JNZ \$+4	
748		JMP ADJOUT	
749		JB7 NEGOFF	
750		MOV R0,#20H	
751		MOV A,@R0	
752		XRL A,#0AH	
753		JZ ADDN	
754		MOV R0,#1EH	
755		MOV A,@R0	
756		JZ \$+8	
757		DEC A	
758		MOV @R0,A	
759	CNTR:	DJNZ R1,\$-7	
760		JMP ADJOUT	
761		INC R0	
762		MOV A,@R0	
763		JZ \$+9	
764		DEC A	
765		MOV @R0,A	
766		DEC R0	
767		MOV @R0,#09H	
768		JMP CNTR	
769		MOV R0,#20H	
770		MOV A,@R0	
771		JZ \$+12	
772		DEC A	
773		MOV @R0,A	
774		DEC R0	
775		MOV @R0,#09H	
776		DEC R0	
777		MOV @R0,#09H	
778		JMP CNTR	
779		MOV @R0,#0AH	
780		JMP ADDN	
781	NEGOFF:	MOV R0,#2 0H	
782		MOV A,@R0	
783		XRL A,#0AH	
784		JNZ CLRB7	
785		MOV R0,#1EH	
786		MOV A,@R0	
787		DEC A	
789		MOV @R0,A	
790		ANL A,#0FH	
791		MOV R1,A	
792		DJNZ R1,\$+4	
793		JMP ADJOUT	
794		MOV R0,#7EH	
795		MOV A,@R0	
796		DEC A	
797		MOV @R0,A	
798		MOV R0,#1EH	
799		MOV A,@R0	
800		JNZ \$+6	
801		MOV R0,#20H	
802		MOV @R0,#00H	
803		JMP NEWOFF	
804	CLRB7:	MOV A,R1	
805		ANL A,#0FH	
806		MOV R1,A	
807	ADDN:	MOV R0,#1EH	

-continued

808		MOV A,@R0	
809		INC A	
810		DA A	
811		NOP	
812		NOP	
813		NOP	
814		JB4 \$+9	
815		MOV @R0,A	
816		DJNZ R1,\$+4	
817		JMP ADJOUT	
818		JMP ADDN	
819		ANL A,#0FH	
820		MOV @R0,A	
821		INC R0	
822		JMP \$-19	
823	ADJOUT:	MOV R0,#1EH	
824		MOV A,@R0	
825		JNZ \$+14	
826		INC R0	
827		MOV A,@R0	
828		JNZ \$+10	
829		INC R0	
830		MOV A,@R0	
831		XRL A,#0AH	
832		JNZ \$+4	
833		MOV @R0,#00H	
834		RETR	
835	RPM:	ANL P2,#0A0H	
836		ORL P2,#01H	
837		MOV R0,#1BH	
838		MOV R7,#03H	
839		MOV R1,#10H	;SET MASK
840	RPMIN:	IN A,P1	
841		MOV R3,A	
842		CPL A	
843		ANL A,R1	
844		XRL A,R1	
845		JNZ RPMIN	
846		MOV A,R3	
847		ANL A,#0FH	
848		MOV @R0,A	
849		INC R0	
850		DJNZ R7,\$+3	
851		RETR	
852		MOV A,R1	
853		RL A	;ROTATE MASK
854		MOV R1,A	
855		JMP RPMIN	
856	COMP:	CLR F1	
857		CLR C	
858		MOV A,R2	
859		CPL A	
860		ADD A,R3	
861		INC A	
862		JNC LTEQ	
863		CPL F1	
864		JN \$+3	
865		CLR F1	
866	LTEQ:	RETR	
867	ENTER:	MOV A,#93H	
868		OUTL P2,A	
869		MOV R0,#18H	
870		MOV A,@R0	
871		INC R0	
872		SWAP A	
873		ADD A,@R0	
874		INC R0	
875		SWAP A	
876		OUTL BUS,A	
877		MOV A,@R0	
878		OUTL BUS,A	
879		CLR A	
880		MOV @R0,A	
881		DEC R0	
882		MOV @R0,A	
883		DEC R0	
884		MOV @R0,A	
885		RETR	
886	MIX:	MOV A,#0F5H	
887		MOV T,A	
888		STRT CNT	;START 10 SEC. TIMER
889		MOV R0,#1AH	

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890      MOV R6,#0DIH
891      MOV R3,#03H
892      MOV A,R6
893      MOVP3 A,@A
894      MOV @R0,A
895      DEC R0
896      INC R6
897      DJNZ R3,$-5
898      CALL ENTER
899      MOV R0,#1DH
900      IRP X, 9BH,11H
901      MOV @R0,#X
902      DEC R0
903      ENDM
908      MOV @R0,#12H
909      CALL DISP
910      MOV A,#82H
911      OUTL P2,A
912      JTF $+11
913      IN A,P1
914      ANL A,#0FH
915      XRL A,#0AH
916      JZ $+4
917      JMP $-9
918      STOP TCNT
919      RETR
920      STODAT:  MOV A,R5
921              MOV R1,A
922              MOV R0,#20H
923              MOV R3,#03H
924              MOV A@R0
925              MOV @R1,A
926              DEC R0
927              DEC R1
928              DJNZ R3,$-4
929              RETR
930      GMESS:  MOV R0,#20H      ;LOAD 'GEL' MESSAGE
931              IRP M, OEH,FH
932              MOV @R0,#M
933              DEC R0
934              ENDM
939              MOV @R0,#10H
940              RETR
941      AUMESS:  MOV R0,#20H      ;LOAD 'AU' MESSAGE
942              IRP M, OBH,OCH
943              MOV @R0,#M
944              DEC R0
945              ENDM
950              MOV @R0,#0DH
951              RETR
952      CLRIF:  PREP 0F0H,15H,0C2H
957              RETR
958              ORG 3D1H
959              DB 07H,00H,00H,06H,00H,00H,03H,00H
960              DB 00H,02H,00H,00H,01H,00H,00H,00H
961              DB 06H,00H,00H,03H,00H,00H,01H,05H
962              DB 00H,00H,06H,3FH,06H,5BH,4FH,66H
963              DB 6DH,7DH,07H,7FH,6FH,40H,00H,77H
964              DB 1CH,3DH,79H,38H,76H,06H,50H
965              END

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Although the presently preferred embodiment of the present invention has been shown and discussed herein, it will be understood that many variations and alternate embodiments of the invention will be apparent to those skilled in the art. For example, other types of optical encoders other than the Gray code encoder may be used to indicate the angular position of the BOB. Furthermore, although the viscometer has been described as being used in connection with drilling fluids in the petroleum industry, it will be understood by those skilled in the art that the invention is applicable as well in measuring rheometric properties of any type of fluid.

What is claimed is:

1. A rotating-cylinder viscometer for measuring the shear stress at a given shear rate of a fluid contained between a rotatable outer cylinder and a rotatable inner cylinder, the inner cylinder rotating in response to

forces produced in the fluid by rotation of the outer cylinder, and where the amount of rotation of the inner cylinder is related to the shear stress of the fluid, the viscometer comprising:

- (a) a system clock, for providing system clock signals;
- (b) a microprocessor unit responsive to said system clock, for inputting and outputting data signals to select the shear rate of rotation of the outer cylinder and to obtain during measurement intervals a plurality of readings of the amount of angular rotation of the inner cylinder at each selected shear rate;
- (c) a shear rate controller responsive to said clock and the shear rate selecting data from said microprocessor, for rotating the outer cylinder at a predetermined constant angular velocity;



- (d) a position indicator connected to the inner cylinder for rotation therewith, for indicating the angular position of the inner cylinder;
- (e) a function selector means connected to said microprocessor, for inputting to said microprocessor the mode of operation for the viscometer, and for inputting the preselected shear rates at which the outer cylinder is to be rotated; and
- (f) display means responsive to said microprocessor, for displaying both a measured shear stress and the angular velocity of the outer cylinder at which the displayed shear stress was measured, said microprocessor determining from the average angular readings for the plurality of angular position readings of the inner cylinder for each measurement interval when the steady state shear stress has been reached, the steady state shear stress obtaining when the average angular readings for consecutive measurement intervals differ by no more than a predetermined amount.
2. The viscometer of claim 1 wherein the system clock is crystal controlled.
3. The viscometer of claim 1 wherein the shear rate controller comprises:
- (a) a motor, for rotating the outer cylinder;
- (b) a speed encoder connected to the outer cylinder for rotation therewith, said encoder outputting a feedback frequency signal;
- (c) a programmable frequency divider responsive to the shear rate selecting data, for dividing the feedback frequency signal to obtain a phase detector clock signal; and
- (d) a motor drive means responsive to the system clock and the phase detector clock signal, for generating the excitation signal to said motor, whereby the motor speed is controlled in accordance with the shear rate selecting data to obtain a constant frequency for the feedback frequency signal.
4. The viscometer of claim 3 wherein said speed encoder is an optical incremental encoder.
5. The viscometer of claim 3 wherein said motor drive means is enabled by said microprocessor to drive said motor, said means comprising:
- (a) a phase detector responsive to a system clock signal and the phase detector clock signal, for generating a phase error signal indicative of the phase between the phase detector clock signal and the clock signal; and
- (b) an amplifier, for generating the motor excitation signal from the phase error signal thereby causing said motor to rotate.
6. The viscometer of claim 3 wherein said motor is a DC-motor of the ironless armature type.
7. The viscometer of claim 3 wherein said programmable frequency divider is a divide-by-N counter.
8. The viscometer of claim 1 wherein said position indicator is an absolute value shaft encoder that outputs a code indicative of the absolute value of the shaft angle.
9. The viscometer of claim 8 wherein the code is a Gray code.
10. The viscometer of claim 9 further comprising a code converter for converting the Gray code to a BCD code.
11. The viscometer of claim 10 wherein said code converter is a read-only memory containing BCD code words that are outputted in response to memory ad-

resses provided by the Gray code output of said absolute value shaft encoder.

12. The viscometer of claims 1 or 8 wherein the predetermined amount of angular difference between the average angular readings of consecutive measurement intervals is  $\pm 1^\circ$ .

13. The viscometer of claim 1 wherein said function selector means comprises:

- (a) a keyboard having a plurality of keys including keys for generating numeric data, for generating digital code words in response to the actuation of the keys; and
- (b) an interface, for interfacing the digital code words to said microprocessor, said interface generating an interrupt strobe when a numeric key is depressed.

14. A method of obtaining the shear stress of a fluid at a selected shear rate speed using a rotating-cylinder viscometer having rotatable inner and outer concentric cylinders comprising the steps of:

- (a) mixing the fluid by rotating the outer cylinder at a high speed;
- (b) rotating the outer cylinder at the selected shear rate speed;
- (c) determining a current average value for the angular position of the inner cylinder from a predetermined number of position measurements taken at a predetermined sample rate, the angular position of the inner cylinder having random fluctuation about a position;
- (d) obtaining the difference between the current average value and the last obtained average value;
- (e) repeating steps (c) through (d) if the difference is greater than a predetermined number of degrees, the current average value becoming the last obtained average value for the next steady state determination; and
- (f) reporting the current average value as the measured shear stress if the difference is less than the predetermined number of degrees.

15. The method of claim 14 wherein the step of mixing the fluid occurs at a speed at least as high as the selected shear rate speed at which the shear stress is to be measured.

16. The method of claim 14 wherein the predetermined sample rate is at least twice the rate of the fluctuation in the angular position of the inner cylinder.

17. The method of claim 16 wherein the predetermined number of degrees is  $\pm 1$  degree.

18. The method of claim 14 further including the steps of:

- (a) determining the angular offset position of the inner cylinder at zero shear rate; and
- (b) correcting all angular position measurements of the inner cylinder by the amount of the zero rate offset.

19. A method of obtaining the steady state shear stress of a fluid at a selected shear rate speed using a rotating-cylinder viscometer in which the fluid is contained between a rotatable outer cylinder and a spring loaded rotatable inner cylinder, the inner cylinder rotating in response to the torque produced by the rotation of the outer cylinder in the fluid whereby the angular rotation of the inner cylinder is indicative of the shear stress of the fluid; the method comprising the steps of:

- (a) mixing the fluid by rotating the outer cylinder at a speed at least as high as the selected shear rate speed at which the shear stress is to be measured for a first predetermined interval of time;



- (b) rotating the outer cylinder at the selected shear rate speed for a second predetermined interval of time;
  - (c) determining a current average value for the angular position of the inner cylinder from a predetermined number of position measurements taken over a third predetermined time interval;
  - (d) obtaining the difference between the current average value and the last obtained average value;
  - (e) repeating steps (b) through (d) if the difference is greater than a predetermined number of degrees, the current average value becoming the last obtained average value for the next steady state determination; and
  - (f) reporting the current average value as the measured shear stress if the difference is less than the predetermined number of degrees.
20. A method of obtaining the shear stress profile of a fluid for various preselected shear rate speeds using a rotating-cylinder viscometer having rotatable inner and outer concentric cylinder comprising the steps of:
- (a) mixing the fluid by rotating the outer cylinder at a high speed;
  - (b) rotating the outer cylinder at a current shear rate speed;
  - (c) determining a current average value for the angular position of the inner cylinder from a position measurements taken at a predetermined sample rate, the angular position of the inner cylinder having random fluctuations about a position;
  - (d) obtaining the difference between the current average value and the last obtained average value, the current average value becoming the last obtained average value for the next difference determination;
  - (e) repeating steps (c) through (d) if the difference is greater than a predetermined number of degrees;
  - (f) storing the current average value as the measured shear stress if the difference is less than the predetermined number of degrees; and
  - (g) repeating steps (c) through (f) until the shear stress at each of the preselected shear rate speeds has been measured thereby obtaining the shear stress profile of the fluid.
21. The method of claim 20 wherein the step of mixing the fluid occurs at a speed at least as high as the highest preselected shear rate speed at which the shear stress is to be measured.
22. The method of claim 20 wherein the predetermined sample rate is at least twice the rate of the fluctuations in the angular position of the inner cylinder.
23. The method of claim 20 wherein the predetermined number of degrees is  $\pm 1$  degree.
24. The method of claim 20 further including the steps of:
- (a) determining the angular offset position of the inner cylinder at zero shear rate; and
  - (b) correcting all angular position measurements of the inner cylinder by the amount of the zero rate offset.
25. The method of claim 23 or 24 wherein the method further includes the steps of:
- (a) rotating the outer cylinder at 3 revolutions per minute;
  - (b) measuring the maximum shear stress reading that occurs during one revolution of the outer cylinder;
  - (c) repeating steps (a) and (b) after a predetermined time delay.

26. A method of obtaining a shear stress profile of a fluid for various preselected shear rate speeds using a rotating-cylinder viscometer in which the fluid is contained between a rotatable outer cylinder and a spring loaded rotatable inner cylinder, the inner cylinder rotating in response to the torque produced by the rotation of the outer cylinder in the fluid whereby the angular rotation of the inner cylinder is indicative of the shear stress of the fluid the method comprising the steps of:
- (a) mixing the fluid by rotating the outer cylinder at a speed at least as high as the highest shear rate speed at which a shear stress is to be measured for a first predetermined interval of time;
  - (b) selecting one of the preselected shear rate speeds as the current shear rate;
  - (c) rotating the outer cylinder at the current shear rate speed for a second predetermined interval of time;
  - (d) determining a current average value for the angular position of the inner cylinder from position measurements taken at a predetermined sample rate over a third predetermined time interval, the angular position of the inner cylinder having random fluctuations about a position over a third predetermined time interval;
  - (e) obtaining the difference between the current average value and the last obtained average value, the current average value becoming the last obtained average value for the next difference determination;
  - (f) repeating steps (c) through (e) if the difference is greater than a predetermined number of degrees;
  - (g) storing the current average value as the measured shear stress if the difference is less than the predetermined number of degrees; and
  - (h) repeating steps (b) through (g) until the shear stress at each of the preselected shear rate speeds has been measured, thereby obtaining the shear stress profile of the fluid.
27. A microprocessor controlled rotating-cylinder viscometer for measuring the steady state shear stress of a fluid located between two coaxially aligned rotatable cylinders in which an outer cylinder is rotated at preselected rates and an inner cylinder is rotated from its zero rate position in response thereto, the viscometer comprising:
- (a) means for rotating the outer cylinder at a constant preselected rate;
  - (b) a shaft encoder connected to the inner cylinder for encoding the angular position of the inner cylinder with respect to the zero rate reference position, the steady state angular position of the inner cylinder for a constant angular rate of the outer cylinder being indicative of the shear stress of the fluid located therebetween; and
  - (c) a programmed microprocessor unit programmed to select the shear rate of rotation of the outer cylinder, and to repetitively obtain at each rate a plurality of readings of the angular position of the inner cylinder during measurement intervals, said processor unit determining the steady state shear stress from average readings of the plurality of angular positions of the inner cylinder determined for each measurement interval, the steady state shear stress obtaining when the average angular positions of the inner cylinder between successive measurement intervals differs by no more than a predetermined value.



28. The viscometer of claim 27 wherein the predetermined value of angular difference is  $\pm 1^\circ$ .

29. The viscometer of claim 27 wherein said shaft encoder is an absolute value shaft encoder for indicating the true angular position of the inner cylinder.

30. The viscometer of claim 27 wherein said rotating means comprises:

- (a) a motor responsive to a driving signal, for rotating the outer cylinder;
- (b) a speed encoder connected to the outer cylinder for rotation therewith, said encoder outputting a feedback frequency signal;
- (c) a programmable frequency divider responsive to the microprocessor, for dividing the feedback fre-

quency signal to obtain a phase detector clock signal;

- (d) a phase detector responsive to said microprocessor and the phase detector signal, for generating a phase error signal indicative of the speed error between the selected shear rate and the actual rate of rotation of the outer cylinder; and
- (e) an amplifier responsive to the phase error signal for generating the drive signal to said motor, whereby the motor speed is controlled in accordance with the selected shear rate from said microprocessor.

31. The viscometer of claim 30 wherein said speed encoder is an optical incremental encoder.

32. The viscometer of claims 30 or 31 wherein said motor is a DC-motor of the ironless armature type.

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