Cross et al.

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[54]	CERAMIC	COMBUSTION LINER
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[58]		arch
		60/39.65
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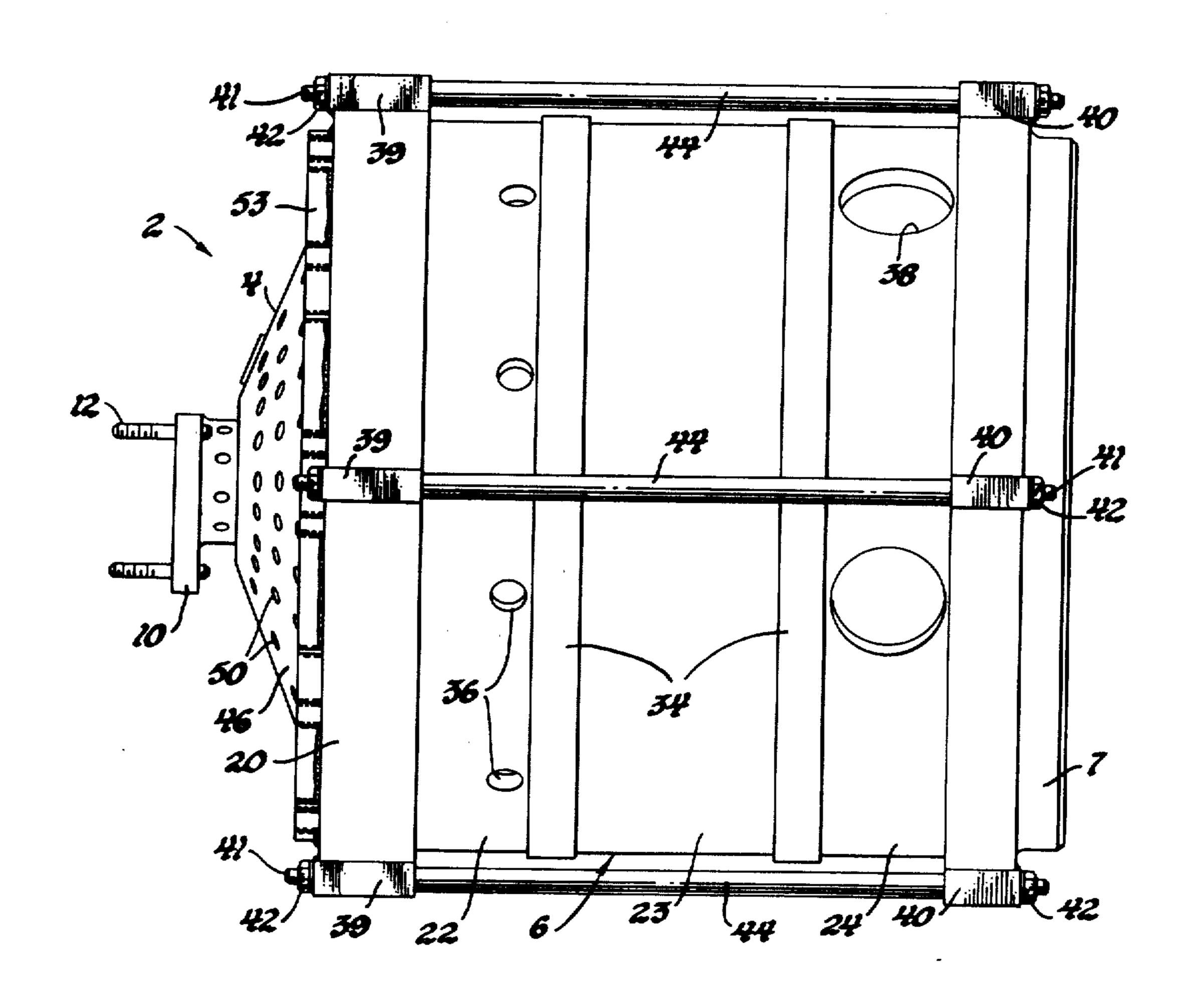
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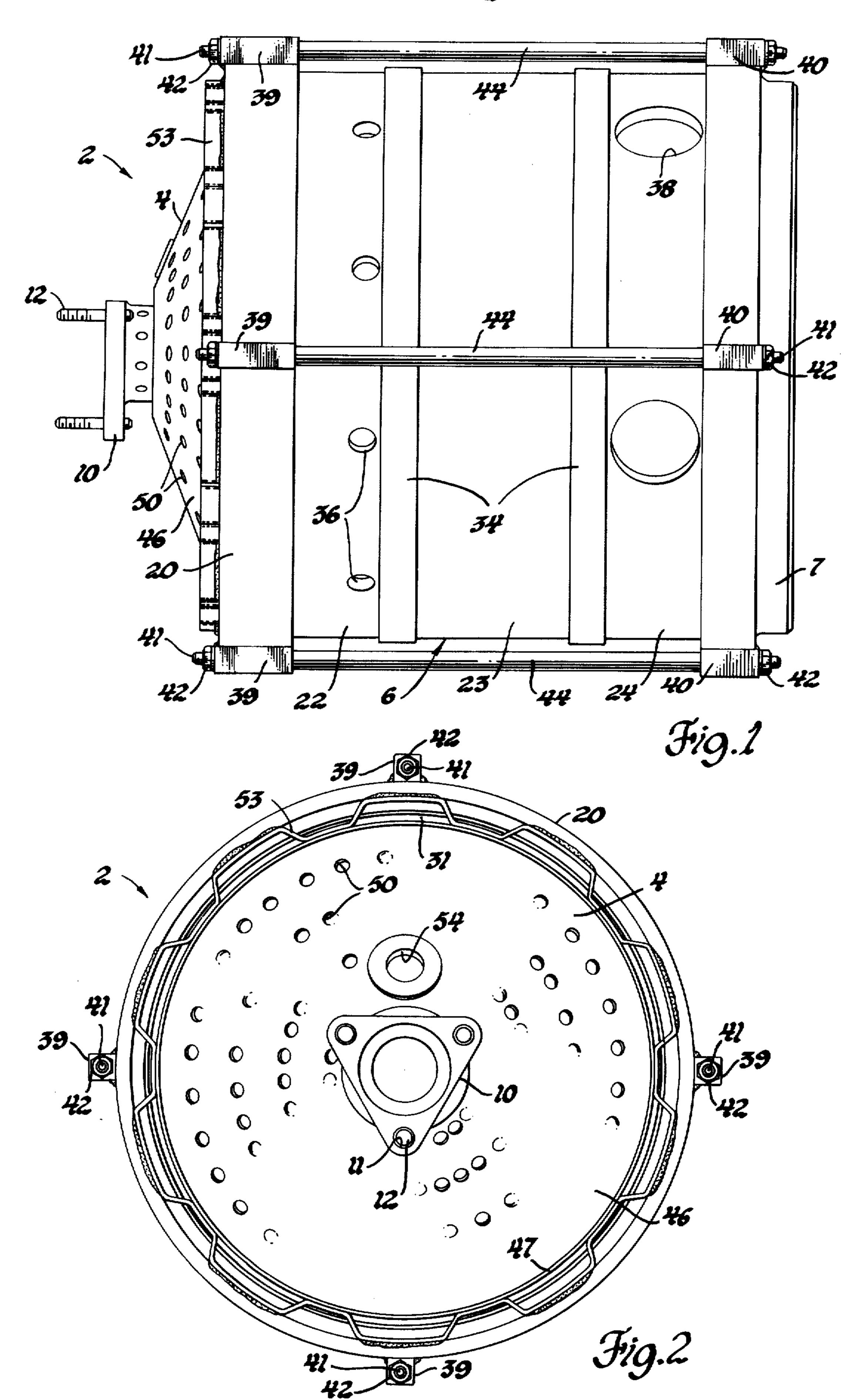
[57] ABSTRACT

A liner for a gas turbine combustion apparatus has a ceramic side wall made up of a number of axially-aligned rings. These are retained between a metal dome and a metal outlet fitting which are connected together by tie rods and maintained at proper axial spacing by sleeves disposed around the tie rods. A T-section metal ring is disposed between each adjacent pair of liner wall segments to align the segments with each other. A resilient compressive connection is disposed between the upstream ring and the dome. These arrangements accommodate differential thermal expansion of the metal and ceramic parts of the liner.

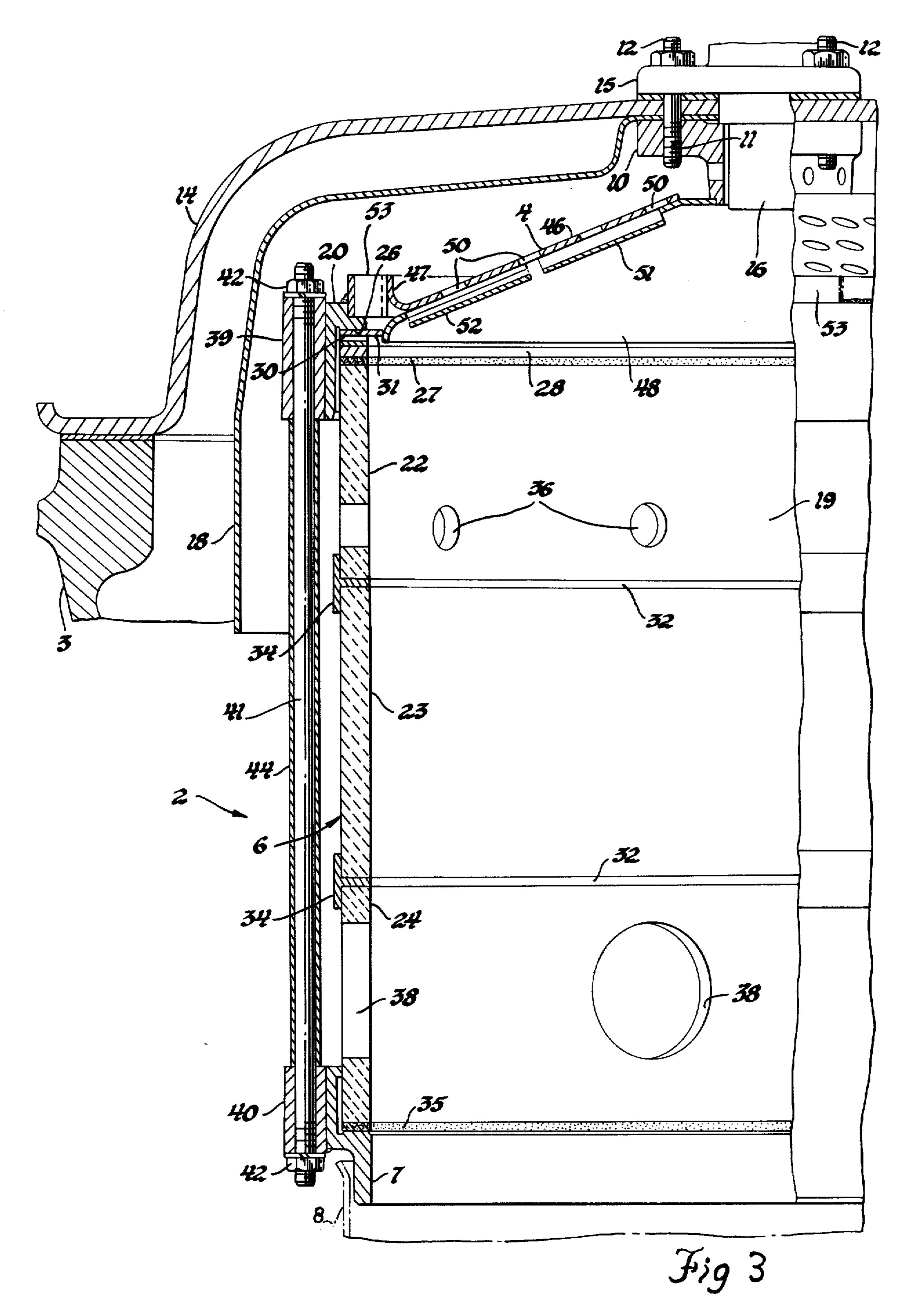
2 Claims, 3 Drawing Figures



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to combustion liner structure for such apparatus.

This invention is directed to combustion apparatus such as is used in a gas turbine engine, and particularly

By way of background, most gas turbine combustion 5 apparatuses include a liner within which combustion takes place. Such liners ordinarily are of circular or annular cross-section, with an upstream end called a dome and an outlet at the downstream end for combustion products. Fuel is sprayed at the upstream end and 10 air enters through the upstream end and through the side wall of the liner to effect combustion and to dilute

the combustion products to a suitable temperature.

Although walls of many combustion apparatuses have been made of various ceramic materials, gas turbine combustion liners in practice, so far as we are aware, have been made of high temperature resisting metal alloys. Such metal alloy structures have good hot strength and a relatively high degree of durability. However, such combustion liners are very expensive; 20 therefore, if ceramics can be substituted for the metal to provide a satisfactory liner, considerable savings may result.

While various known ceramics are highly resistant to heat and may be formed into cylinders and other 25 shapes by known techniques, such materials are relatively weak and brittle. Also, the ceramics have relatively low thermal expansion, which presents a problem when it becomes necessary to mount them with metal components in an engine combustion apparatus.

This invention is directed to a structure which facilitates the employment of ceramic elements for the major portion of a combustion liner so as to retain the cost and temperature resisting advantages of the ceramic while avoiding stresses on the ceramic material which would be likely to cause cracking or breaking.

Briefly stated, the combustion liner is made up of a metal dome or upstream end cap which may be similar to previously known structues, a number of aligned ceramic rings defining the side wall of the combustion liner, an outlet fitting engaging the farthest downstream ring, means between the rings to align them, and an interconnection between the dome and the outlet fitting providing a confining cage or support for the liner wall sections. These structures embody an arrangement to accommodate the relatively greater expansion of the metal parts than the ceramic.

The principal objects of the invention are to provide a combustion liner of largely ceramic composition which is adpated to withstand the shocks, both mechanical and thermal, which are encountered in service, to provide an economical and durable combustion liner largely of ceramic composition, and to provide a simple arrangement for supporting a ceramic combustion liner in a gas turbine engine or other combustion apparatus.

The nature of the invention and its advantages will be more clearly apparent to those skilled in the art from the succeeding detailed description of the preferred embodiment of the invention and the accompanying drawings thereof.

FIG. 1 is a side elevation view of a combustion liner, shown with its axis horizontal.

FIG. 2 is an end elevation view of the same.

FIG. 3 is a longitudinal sectional view of the liner as installed with its axis vertical in a gas turbine engine combustion apparatus.

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FIG. 3 is a partial view of the liner as installed in a gas turbine engine of known type which may be similar to those described in United States patents as follows: Collman et al. U.S. Pat. No. 3,077,074, Feb. 12, 1963; Collman et al. U.S. Pat. No. 3,267,674, Aug. 23, 1966; and Bell U.S. Pat. No. 3,490,746, Jan. 20, 1970. The combustion liner 2 may be mounted in a suitable space within the engine, a portion of the frame or housing of the engine being designated at 3. The combustion liner includes a metal dome or upstream end cap 4 and a side wall 6 which preferably is of circular cross section, as illustrated. The downstream end of the combustion liner is defined by a ring-shaped outlet fitting 7 which may telescope into a scroll or transition section 8 which conducts the combustion products from the combustion liner into the turbine of the engine (not illustrated).

The dome 4 includes a central hollow flanged bracket 10. The flange is of triangular outline and has three threaded holes 11 to receive studs 12 by which the dome is mounted in the engine so as to support the liner. Studs 12 extend through a combustion cover 14 suitably fixed to the frame 3 of the engine. The studs extend through a flange 15 on a fuel spray nozzle 16. Nuts on the studs thus hold the nozzle against the outer surface of the cover 14 and the flange 10 against the inner surface. A heat shield 18, which shields the combustion cover 14 from heat radiated within the engine and influences air flow to the dome, is also mounted between the flange 10 and cover 14. The dome 4 may be of structure previously known providing for admission of a large part of the combustion air through the dome into a combustion space 19 defined within the liner. Passing over details of the dome for the time being, it may be noted that this is a sheet metal structure to the outer margin of which is fixed a mounting ring 20.

The liner wall 5 illustrated is composed of three annular sections or rings 22, 23, and 24 in the direction of flow from the dome to the outlet fitting 7. This segmented wall allows for different expansion of the sections of the liner wall while minimizing stress. In operation, the central section 23 becomes much hotter than the adjoining sections. Since they are separate, they do not stress each other because of differential expansion.

Wall section 22 fits with some slight clearance within the mounting ring 20. The end of section 22 abuts through intermediate structure against a flange 26 on the mounting ring 20. The intermediate structure shown includes a slightly yieldable ceramic felt washer or gasket 27 of commercially available material, a rigid washer 28 of heat resistant metal, a wavy resilient washer or annular spring 30 of heat resistant metal alloy, and an air flow controlling or blocking washer 31. The felt washer 27 may be omitted, since the rigid washer 28 distributes the concentrated force from the wave washer convolutions over the end surface of the liner section 22.

An aligning ring 32 is disposed between each two adjacent wall sections 22 and 23 or 23 and 24. These rings, which are of T-shaped cross section, include an outer band 34 which engages the outer surface of the wall rings to align them and a portion which extends between the end faces of adajcent wall rings.

A second yieldable ceramic washer 35 may be disposed between the downstream end of wall section 24 and the outlet fitting 7.

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In the embodiment illustrated, the upstream wall section 22 has a ring of ten primary air entrance holes 36 and the downstream wall section 24 has five equally spaced secondary or dilution air holes 38. Thus, so far as the air entrances through the side wall is concerned, the liner is similar to a common type of metal-walled liner. Any suitable arrangement of air holes may be used.

The several parts of the liner are maintained in assembled relation by structure including sleeves 39 fixed 10 90° apart around the circumference of mounting ring 20 and sleeves 40 fixed 90° apart around the outlet fitting 7. These sleeves may be welded to the parts 20 and 7 respectively. Tie rods 41 extending through the aligned sleeves 39 and 40 mount nuts 42 on the 15 threaded ends of the rods which bear through washers against the outer ends of sleeves 39 and 40.

The spacing between parts 20 and 7 is controlled by hollow struts 44 which are fitted over the tie rods 41 before the assembly is completed. Thus, by tightening 20 the nuts 42, the sleeves 39 and 40 are brought to bear against the ends of the struts, definitely establishing the spacing between the flange 26 of mounting ring 20 and the bearing surface of the outlet fitting 7. This distance should be such as to compress to some extent the wash- 25 ers 27 and 35 (if provided) and the wavy spring 30. When the apparatus heats up during combustion, the metal parts such as the tie rods and sleeves shown, or other metal supporting structures for the liner, will ordinarily expand somewhat more than the ceramic, not- 30 withstanding their lower temperature. The preloaded spring 30 will then expand resiliently to prevent looseness of the parts which might allow shock or vibration to cause chipping of the ceramic.

The dome 4 illustrated is an adaptation of the dome 35 structure used on a prior all-metal combustion liner. Any suitable dome structure may be used. This one is effectively a double-walled structure with an outer layer 46 terminating in a flange 47 at its outer edge and an inner layer 48 which is fixed by welding or brazing 40 to the bracket 10. Numerous primary air holes 50 extend through the outer layer 46. The inner layer 48 is deformed to provide louvers 51 and 52 which swirl the air entering through the holes 50 in opposite directions.

The flange 47 is welded to a zigzag strip 53 which in 45 turn is welded to the mounting ring 20. The original purpose of the zigzag strip is to admit film cooling air to the interior of the liner. In a ceramic liner structure such as that described here, there is no need, or only a very slight need, for such cooling air. To control the 50 flow of air through the zigzag strip into the interior of the liner, the washer 31 extends across the gap between the ring 20 and the layer 48 of the dome. The amount of air flow can be varied by choice of width of washer 31. The dome also provides an opening 54 (FIG. 2) for 55 an igniter.

The operation of the combustion chamber is similar to that of other combustion chambers of like overall configuration. Air entering through the holes 50 and 36 provides for combustion of fuel sprayed through the 60 nozzle 16. The resulting combustion products are diluted by additional air entering through openings 38 and the resulting mixture is discharged through the outlet fitting 7 and into the turbine through suitable structure. It will be seen that the liner structure described 65 provides for accurate alignment and support of the ce-

ramic parts without concentrated loads or freedom for vibration and rattling of the parts. It thus is adapted to promote long life of the ceramic notwithstanding its lack of strength and its brittleness. The mounting structure is simple and may readily be taken apart for replacement of any part which deteriorates in service.

The nature of the ceramic is a matter of choice. So far it appears that a silicon carbide ceramic is best adapted to meet the requirements of practice. The metal parts need have no unusual property except sufficient strength at high temperature and resistance to oxidation or corrosion to have sufficient life in service. The tie rods 41 and struts 44 are out of direct contact with the hot ceramic parts which are the hottest parts of the liner.

By segmenting the ceramic, it is possible to minimize thermal stresses due to uneven heating of the ceramic because of the differences in temperature at various zones of the combustion liner.

For information, it may be pointed out that the liner described is intended to operate at temperatures up to 1057°F. for inlet air and up to 1763°F. for outlet air temperature to the turbine. It operates at about five atmospheres with an air flow of about 4½ lbs. per second and a fuel flow of about 190 lbs. per hour at full capacity.

It should be apparent to those skilled in the art from the foregoing that the combustion liner structure described is particularly well adapted to meet the requirements of practice in a small gas turbine engine.

The detailed description of the preferred embodiment of the invention for the purpose of explaining the principles thereof is not to be considered as limiting or restricting the invention, since many modifications may be made by the exercise of skill in the art.

We claim:

- 1. A combustion liner for a combustion apparatus comprising, in combination, a dome defining the upstream wall of the liner; a plural number of alinged ceramic wall rings defining the side wall of the liner, one ring being adjacent to the dome; means for aligning the said one ring with the dome; an aligning ring disposed at each junction between adjacent wall rings engaging a radially facing surface of each wall ring to align the wall rings; an outlet fitting abutting the farthest downstream wall ring; and means interconnecting the dome and the outlet fitting including tie rods distributed circumferentially of the liner between the dome and outlet fitting.
- 2. A combustion liner for a combustion apparatus comprising, in combination, a dome defining the upstream wall of the liner and providing support means for the liner; a plural number of aligned ceramic wall rings defining the side wall of the liner, one ring being adjacent to the dome; means for aligning the said one ring with the dome; an aligning ring disposed between each two adjacent wall rings engaging an end surface and a radially facing surface of each wall ring to align the wall rings and transmit compressive force between the wall rings; an outlet fitting abutting the farthest downstream wall ring; and means independent of the wall rings interconnecting the dome and the outlet fitting including tie rods distributed cirumferentially of the liner between the dome and outlet fitting.