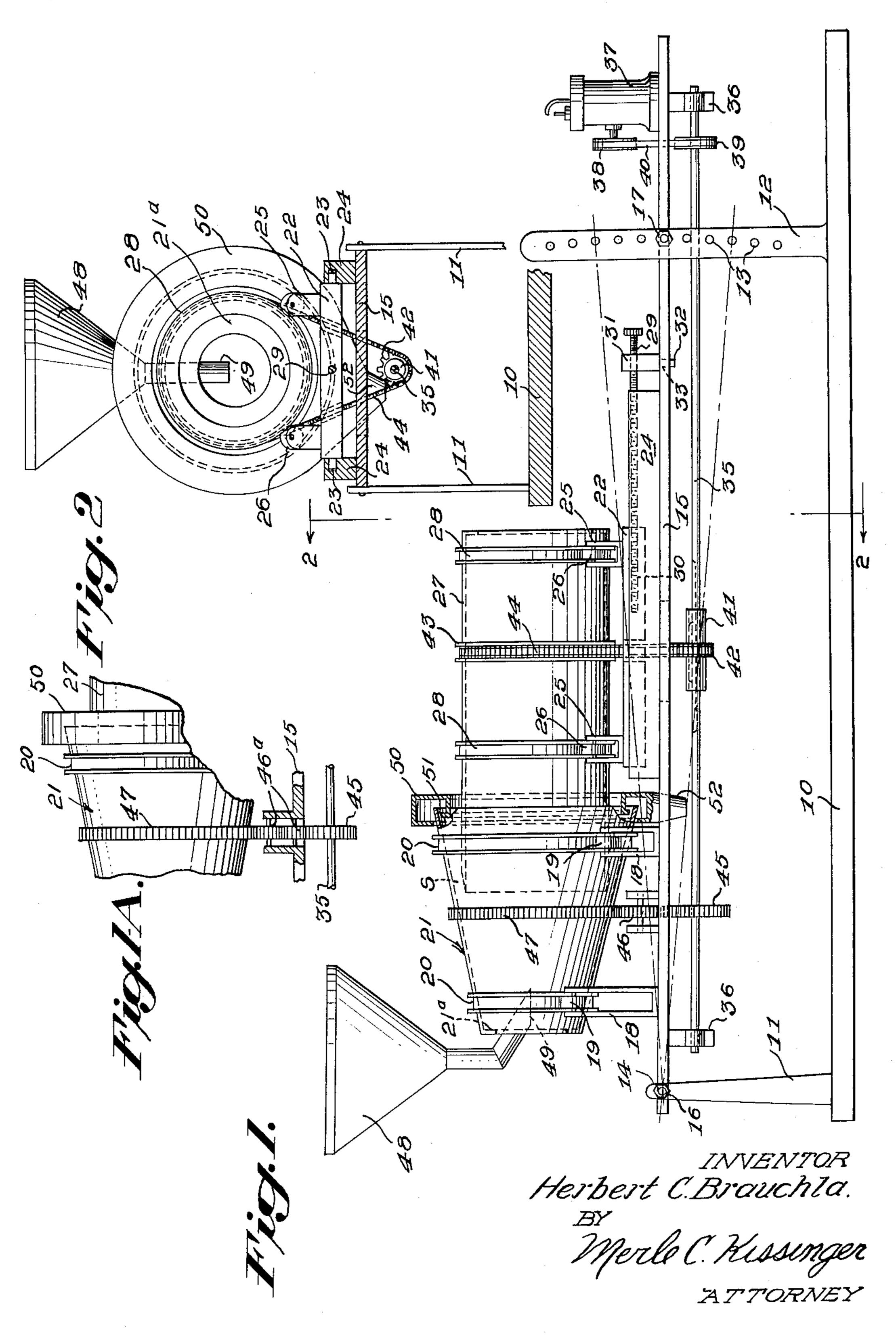
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DIFFERENT SPECIFIC GRAVITIES

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2 Sheets-Sheet 1



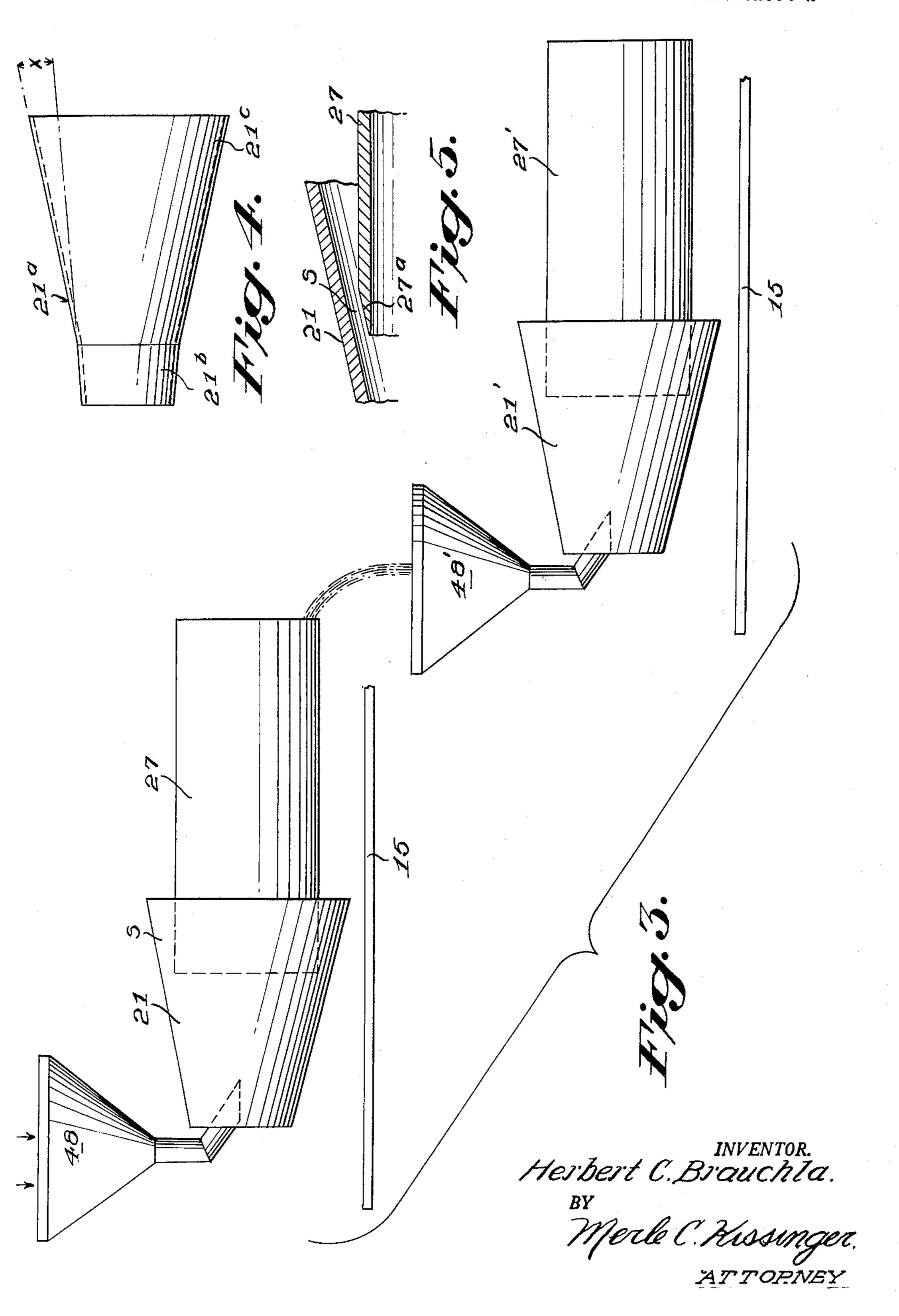
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2,995,247 APPARATUS FOR SEPARATING MATERIALS OF DIFFERENT SPECIFIC GRAVITIES Herbert C. Brauchla, 2300 Buckland Ave., Fremont, Ohio Filed Mar. 9, 1960, Ser. No. 13,756 4 Claims. (Cl. 209—482)

This invention relates to apparatus for separating materials of different specific gravities.

A primary object of the invention is an improved appa- 10 ratus for separating materials of different specific gravities in a conglomerate mass thereof, wherein the mass is confined to rotation thereof about an axis and wherein a portion of the material of a certain specific gravity is separated from the remaining material which is subjected 15 to rotation about said axis and wherein the materials of different specific gravities are collected at separate stations.

A further object of the invention is the provision of means for separating materials of different specific gravities which embodies a pair of rotatable co-axial drums 20 with an end of one thereof disposed within an adjacent end of the other and wherein said drums are rotatable about said axis with provision for feeding a heterogeneous mass of material into one end of one of the drums and with discharge of material of a given specific gravity from 25 one of the drums and the remaining material discharging from the other drum.

A further object of the invention is a relatively simple structure for effective separation of materials of different specific gravities and which is highly efficient in operation 30 as well as relatively simple in construction.

Other objects and advantages of the invention will become apparent in the course of the following detailed description, taken in connection with the accompanying drawings, wherein:

FIG. 1 is a side elevational view of the improved material separating apparatus.

FIG. 1-A is a fragmental side elevational view corresponding to a portion of FIG. 1, but disclosing a modified embodiment of the invention.

FIG. 2 is a broken transverse vertical sectional view as observed in the plane of line 2—2 on FIG. 1.

FIG. 3 is a more or less diagrammatic side elevational view showing material separating apparatus according to FIG. 1 as being arranged in tandem.

FIG. 4 is a side elevational view of a modified form of the advance separating member, shown in FIG. 1.

FIG. 5 is a fragmental sectional view of adjacent portions of the two cooperating members embodied in the separating apparatus.

Referring now in detail to the drawings, and first to FIGS. 1 and 2 thereof, the improved structure will be seen to comprise a frame including a base member 10 from one end of which a pair of laterally opposed posts 11 project vertically upwardly and adjacent the opposite end of the base member 10 are a pair of laterally opposed supports 12 which are provided with vertically spaced apertures 13 and, as is shown in FIG. 1, the apertures in each support have their centers in a curved line concentric with apertures 14 in the upper ends of the posts 11.

Furthermore, the apertures 13 in the supports 12 are transversely aligned.

A platform 15 has one end thereof pivotally supported between the posts 11, as by means of a pin 16 and the opposite end of the platform 15 is releasably engaged by 65 pins 17 which are selectively receivable within the apertures 13 whereby the platform 15, shown in horizontal position, is capable of being disposed in different degrees of angular relation to the base member 10, as is indicated by dot-and-dash lines.

Disposed on the upper face of the platform 15 are brackets 18 arranged in longitudinally spaced pairs and in

which are rotatably supported rollers 19 which are operatively disposed within channel tracks 20, disposed adjacent opposite ends of a frusto-conical drum 21 and the cooperating rollers 19 and tracks 20 provide for rotation of the drum 21 about the axis thereof.

At this point, it is to be observed that the cooperating channel tracks 20 and the rollers 19 maintain the drum 21 against movement parallel with its axis and it is to be further observed that the drum 21 at the smaller or feed end thereof is provided with a relatively shallow circular flange 21a.

A carriage 22 is provided with laterally opposed cam followers 23 which are disposed within tracks 24 on the platform 15. The cam followers in cooperation with the tracks provide for relative lateral adjustment of the drums for accurate axial alignment thereof.

Projecting upwardly from the carriage 22 adjacent each end thereof are a pair of laterally spaced brackets 25 in each of which is rotatably supported a roller 26.

A cylindrical drum 27 is provided with a pair of circumferential channel tracks 28 in substantially spaced relation axially of the drum in which the rollers 26 are disposed whereby the drum is capable of rotation about its axis which is normally co-incident with the axis of the frusto-conical drum 21.

The forward end of the drum 27 is normally telescoped within the rear and larger end of the frusto-conical drum 21, but same is capable of axial adjustment with a corresponding change in spacing of the inner end of the drum 27 and the inner wall of the drum 21 for a purpose later to appear.

In order to vary the distance of projection of the forward end of drum 27 within the drum 21, the carriage 22 is capable of longitudinal adjustment and which is 35 effected by means of an elongated screw 29 having one end thereof adjustably disposed within a threaded bore 30 in the rear end of the carriage 22 and the screw is threaded through a block 31 having a reduced square extension removably disposed in an aperture 33 in the plat-40 form **15**.

Thus, upon turning the screw 29, the drum 27 may be axially adjusted for varying the space S between its inner end and the wall of the frusto-conical drum 21.

Furthermore, the screw 29 may be threaded entirely 45 out of bore 30 and same, together with block 31, may be laid aside and the drum moved to a position wherein it is entirely clear of drum 21 for cleaning thereof.

Furthermore, the cam followers 23 provide for critical adjustment of the two drums 21 and 27, for efficient operation thereof.

The drums 21 and 27 are rotatable about their common axis and according to FIG. 1, the drum 21 is rotatable at a higher rate of speed than drum 27 and in the same direction which is effected by drive means comprising an elongated shaft 35 whose opposite ends are rotatably supported in bearings 36 depending from the platform 15. The shaft 35 is driven by a motor 37 through pulleys 38 and 39 and a belt 40.

A cylindrical sleeve 41 surrounds shaft 35 and has a 60 spline connection therewith and a sprocket wheel 42 surrounds and is fixed to the sleeve and the drum 27 is provided with a sprocket wheel 43 disposed intermediate tracks 28 and which is drivingly engaged with the sprocket wheel 42 by means of a sprocket chain 44.

The purpose of the splined sleeve 41 is to provide for axial adjustment of the drum 27 and upon adjustment of the drum 27 to a cleaning position and in retracted relation to drum 21, the chain 44 may be temporarily removed.

A relatively large gear 45 is secured to shaft 35 and rotatably supported on the platform 15 is a relatively small gear 46 in driving engagement with gear 45 and

which is also in driving engagement with a gear 47 surrounding the frusto-conical drum 21 intermediate tracks 20. Accordingly, the drum 21 will rotate faster than and in the same direction as drum 27.

According to FIG. 1-A the drum 21 is rotated faster 5 than and in an opposite direction to drum 27 which is effected by a pair of meshing relatively small gears 46° which are also in mesh with gears 45 and 47.

A material receiving hopper 48 has a spout 49 leading into the smaller and feed end of the drum 21 and a 10 circular housing 50 is provided for receiving higher specific gravity material which passes between the inner wall of drum 21 and the outer wall of the inner end of drum 27, or through space S.

The housing 50 is fixed with adjacent ends of drums 15 21 and 27 rotating within it and such housing is of generally rectangular form in cross section with the larger end of drum 21 entering a circular opening 51 therein and the lowermost portion of the housing is of funnel-like formation as indicated at 52, which is disposed to one side 20 of shaft 35 for discharge of material without interruption of separating operations.

In FIG. 3 is shown in more or less diagrammatic fashion a tandem arrangement wherein material discharged from drum 27 enters a hopper 48¹ of a second unit having 25 cooperating drums 21¹ and 27¹ whereby the valuable material is capable of more complete extraction from an original heterogeneous mass of material.

In FIG. 5 is shown a desirable formation of the inner end of drum 27 which is bevelled at 27^a to present a sur- 30 face parallel with the inner wall of drum 21 for most effective adjustment of the space S which may be rendered zero.

As shown in FIG. 4, the drum 21^a may have a forward end portion 21^b whose wall is at a less angle to the axis 35 of the drum than the major portion 21^c thereof. In this form of drum, the material from the hopper spout 49 would be deposited on the portion 21^b.

Having set forth the invention in accordance with certain specific structural embodiments thereof, what is 40

claimed and desired to be secured by U.S. Letters Patent is:

1. Apparatus for separating materials of different specific gravities, comprising a vertically adjustable platform, a frusto-conical drum supported on said platform for rotation about its axis, a material feed hopper extending into the smaller end of said frusto-conical drum, a cylindrical drum disposed about said platform in normally co-axial relation to said frusto-conical drum and having one end thereof normally disposed within said frusto-conical drum, means on said platform longitudinally adjustable relative thereto, means on said last means supporting said cylindrical drum for rotation about its axis, a power driven shaft disposed beneath said platform and means for individually imparting rotation to said drums from said power driven shaft.

2. The structure according to claim 1, wherein said frusto-conical drum is driven at a higher rate of speed than said cylindrical drum.

3. The structure according to claim 1, together with a housing communicating with the larger end of said frustoconical drum and extending circumferentially thereof, said housing being adapted to receive material discharged from said frusto-conical drum and having a discharge funnel-like member at the lowermost portion thereof.

4. The structure according to claim 1, wherein said drums are rotated in opposite directions.

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