



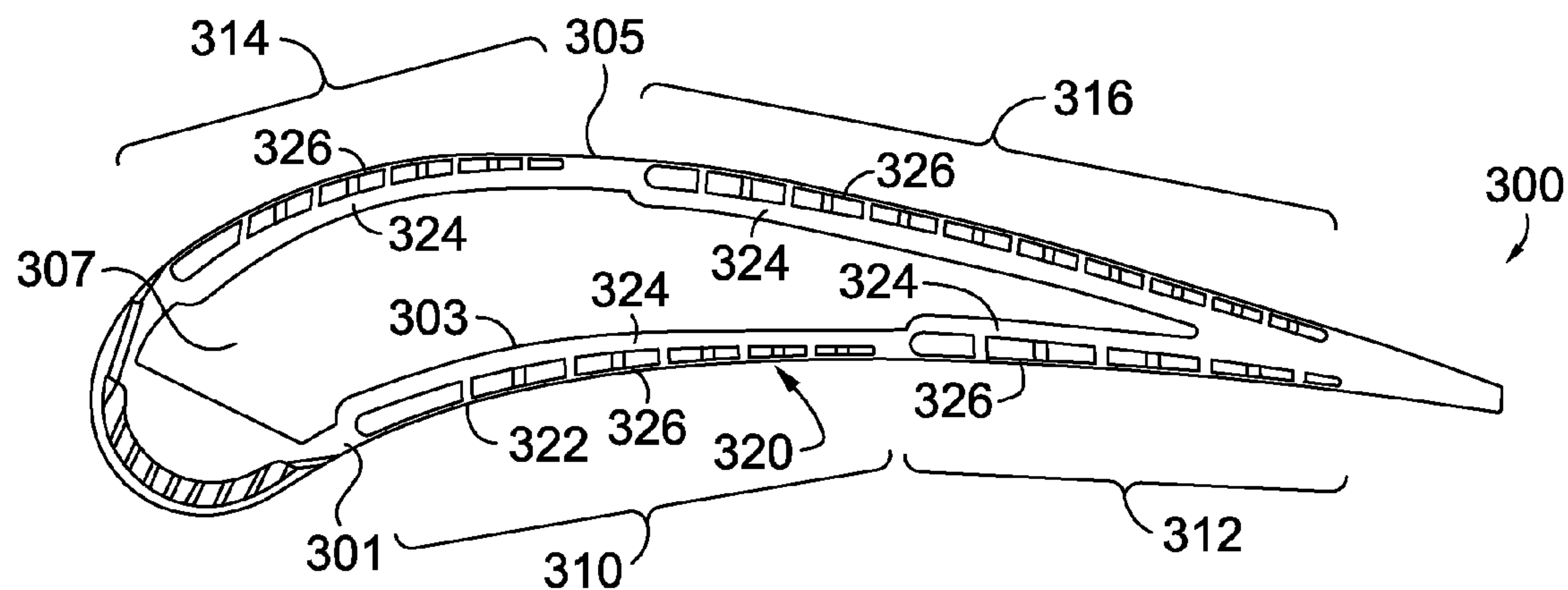
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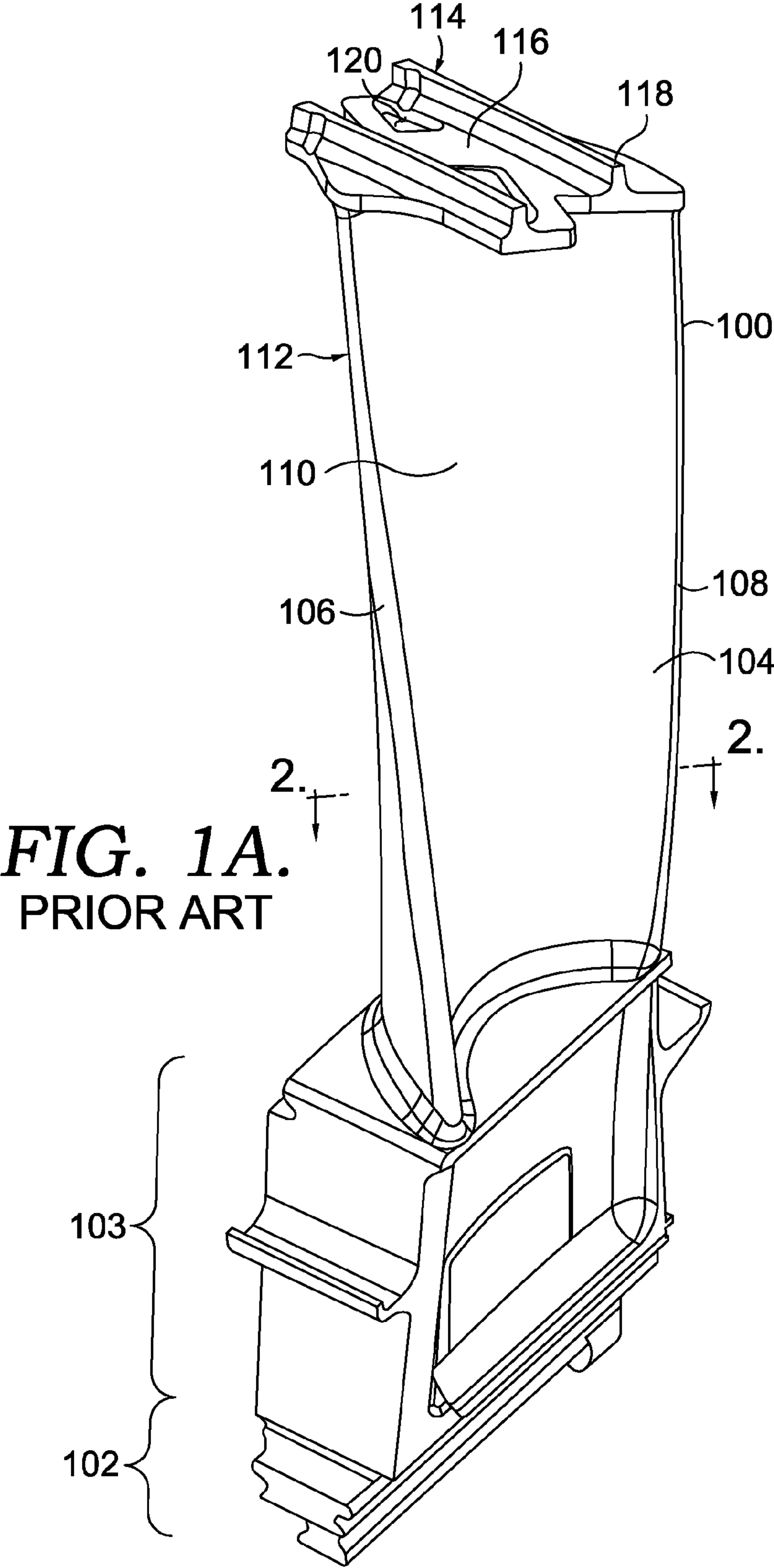
(19) **United States**(12) **Patent Application Publication**
PIZANO et al.(10) **Pub. No.: US 2016/0146019 A1**(43) **Pub. Date: May 26, 2016**(54) **COOLING CHANNEL FOR AIRFOIL WITH
TAPERED POCKET****Publication Classification**(71) Applicants: **Elena P. PIZANO**, Golden, CO (US);
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(2013.01); **F01D 9/041** (2013.01); **F01D 25/12**
(2013.01); **F05D 2220/32** (2013.01); **F05D**
2260/202 (2013.01); **F05D 2240/12** (2013.01);
F05D 2240/30 (2013.01); **F05D 2230/31**
(2013.01); **F05D 2260/22141** (2013.01)(21) Appl. No.: **14/951,163**(22) Filed: **Nov. 24, 2015****Related U.S. Application Data**(60) Provisional application No. 62/084,810, filed on Nov.
26, 2014.

(57)

ABSTRACT

The present invention includes systems and methods for providing cooling channels located within walls of a turbine airfoil. These cooling channels include micro-circuits that taper in various directions along the length and width of the airfoil. In addition, these cooling channels have a variety of shapes and areas to facilitate convective heat transfer between the surrounding air and the airfoil.





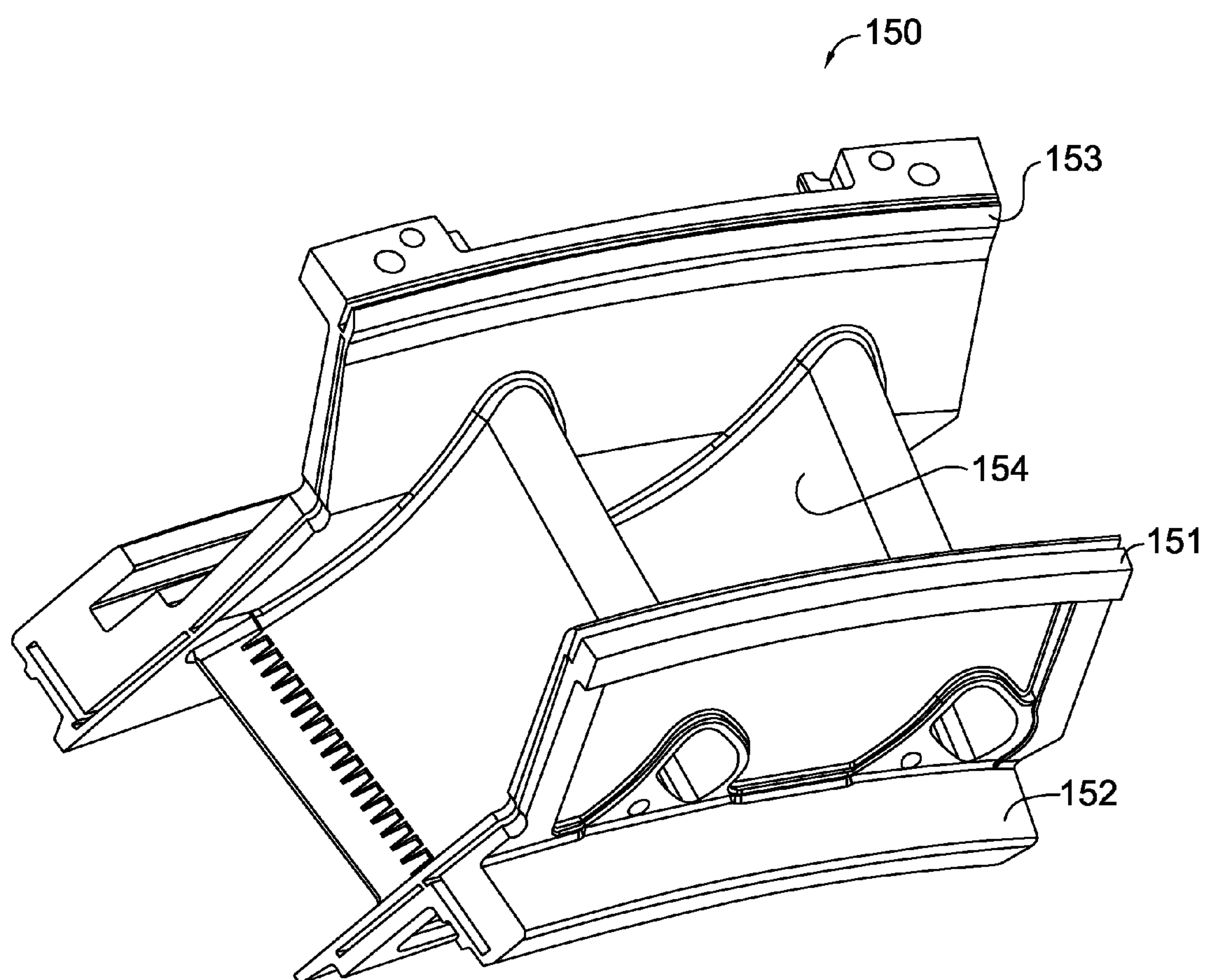


FIG. 1B.
PRIOR ART

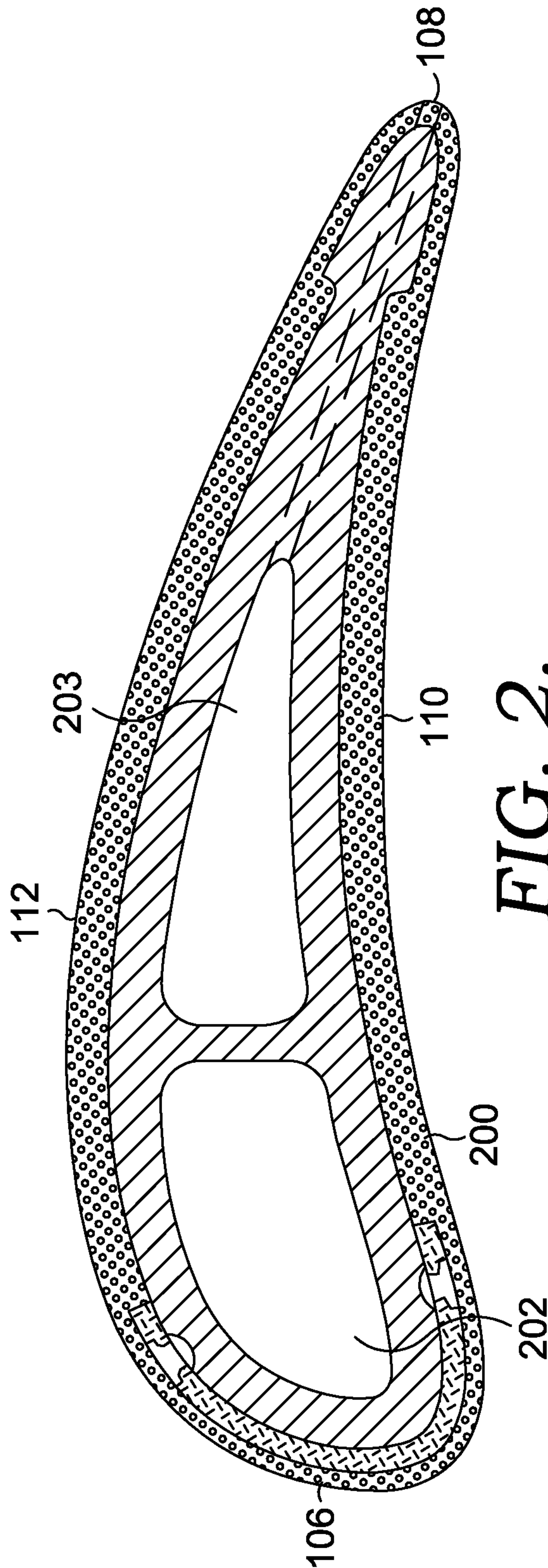
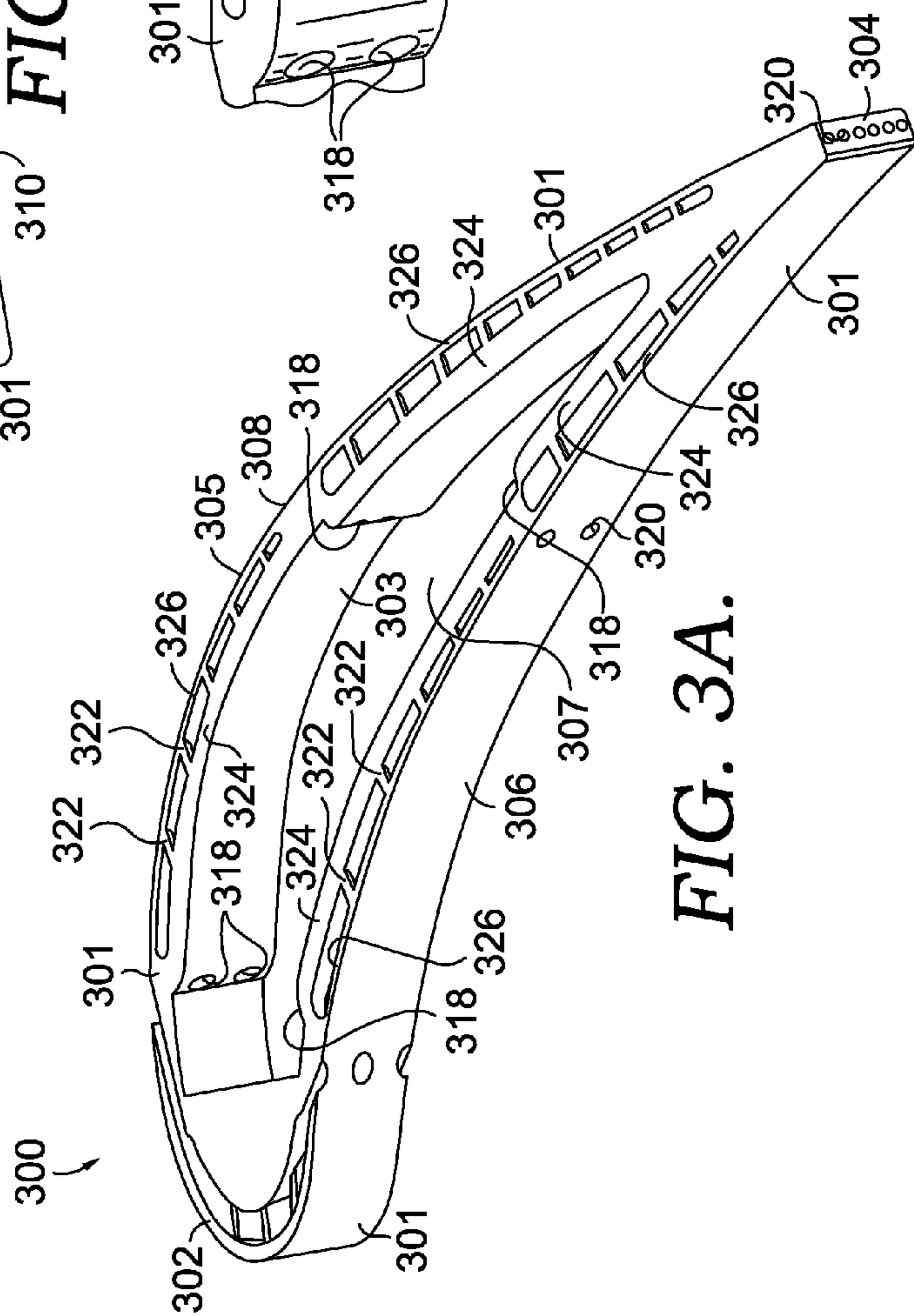
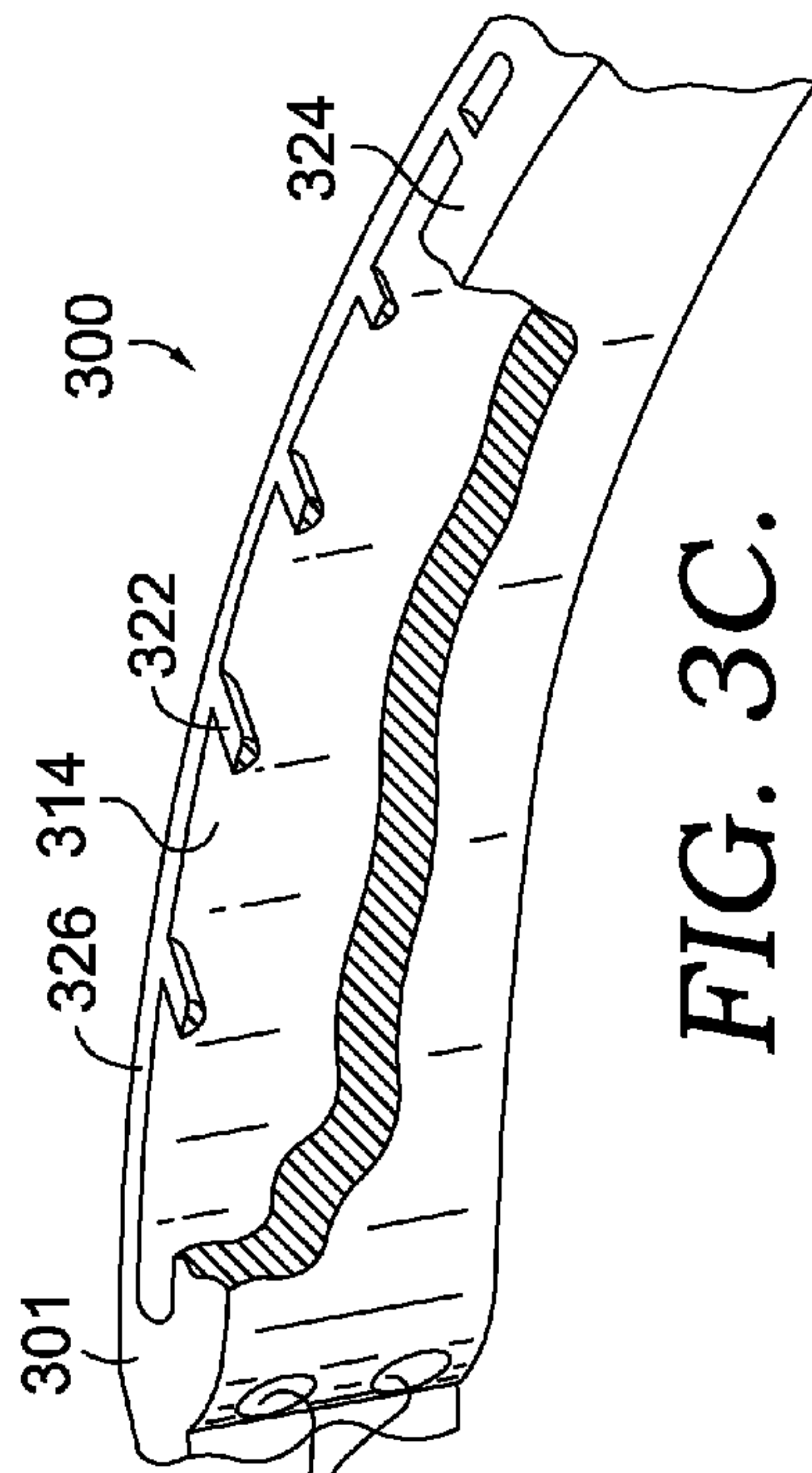
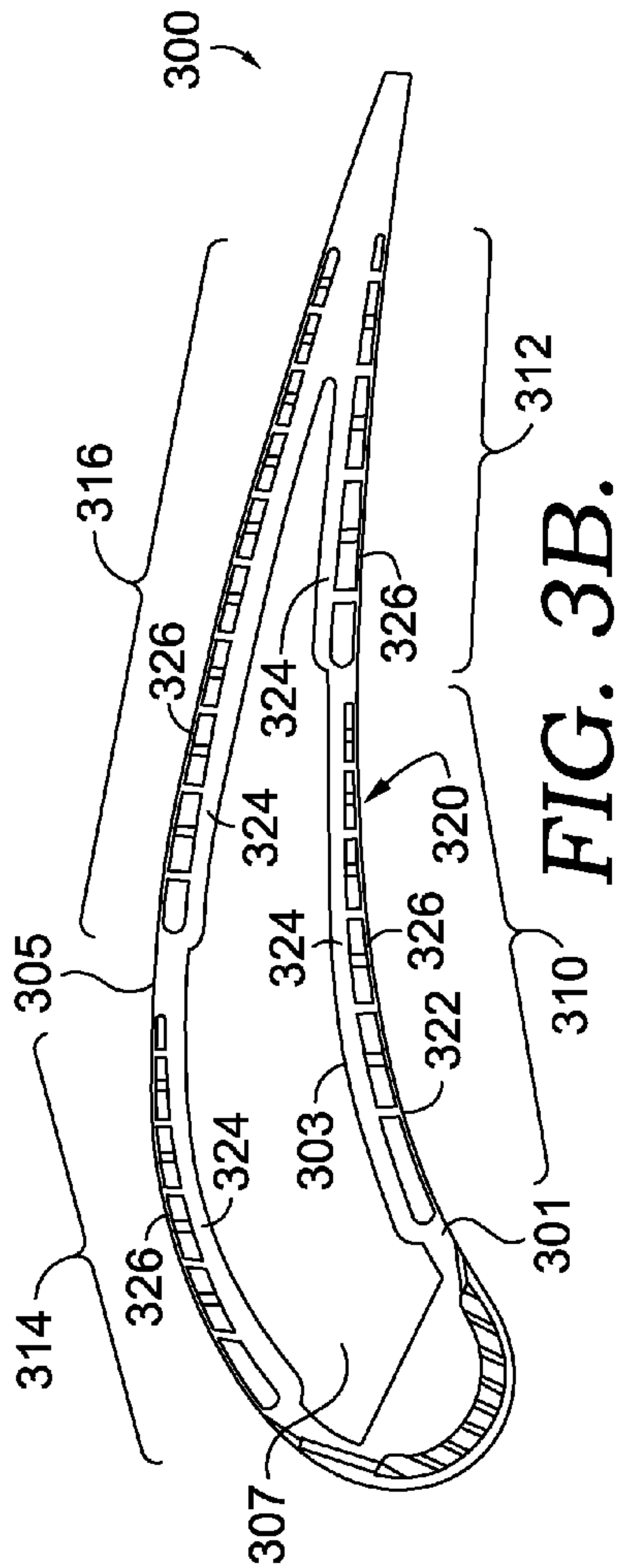
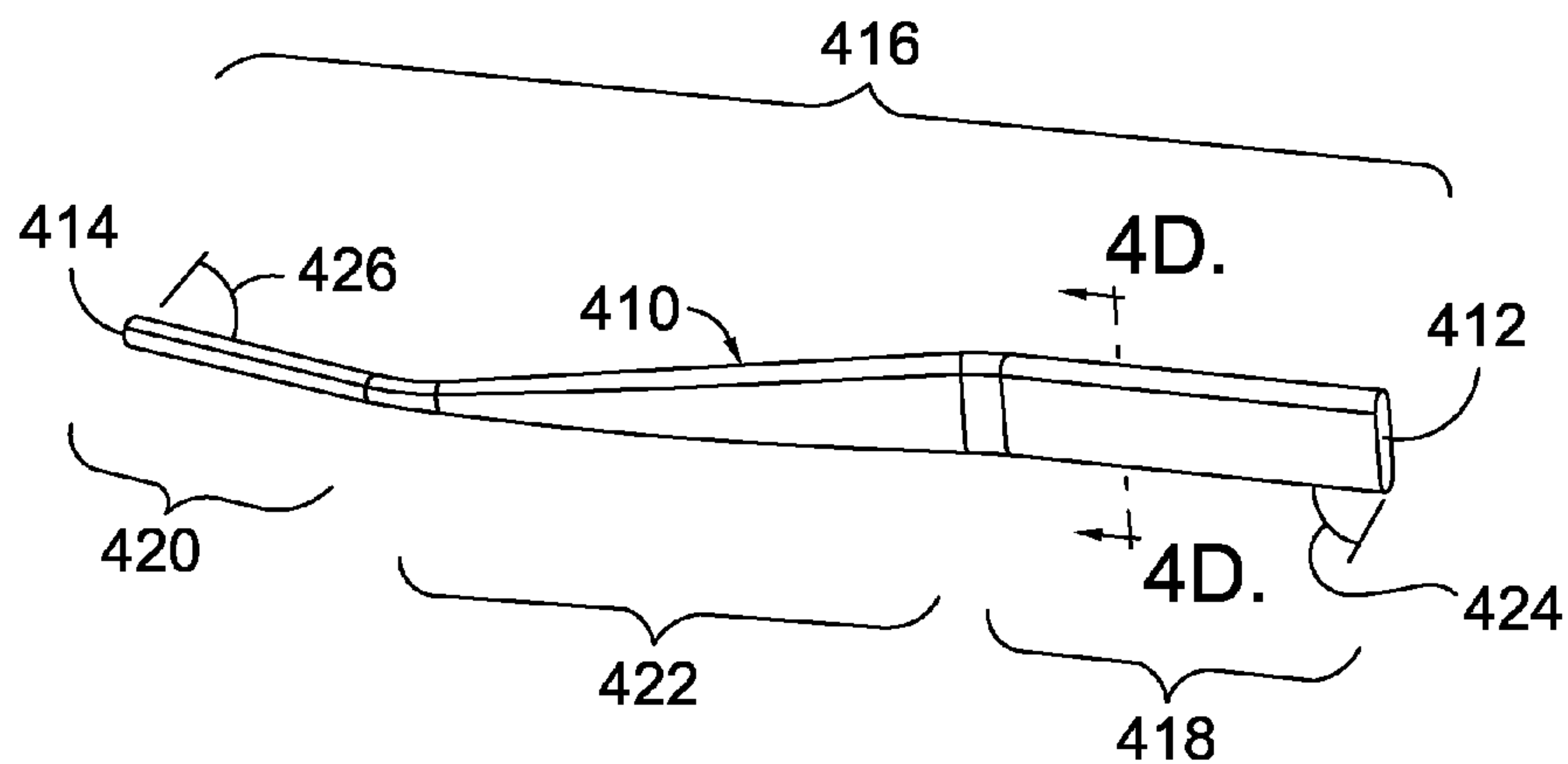
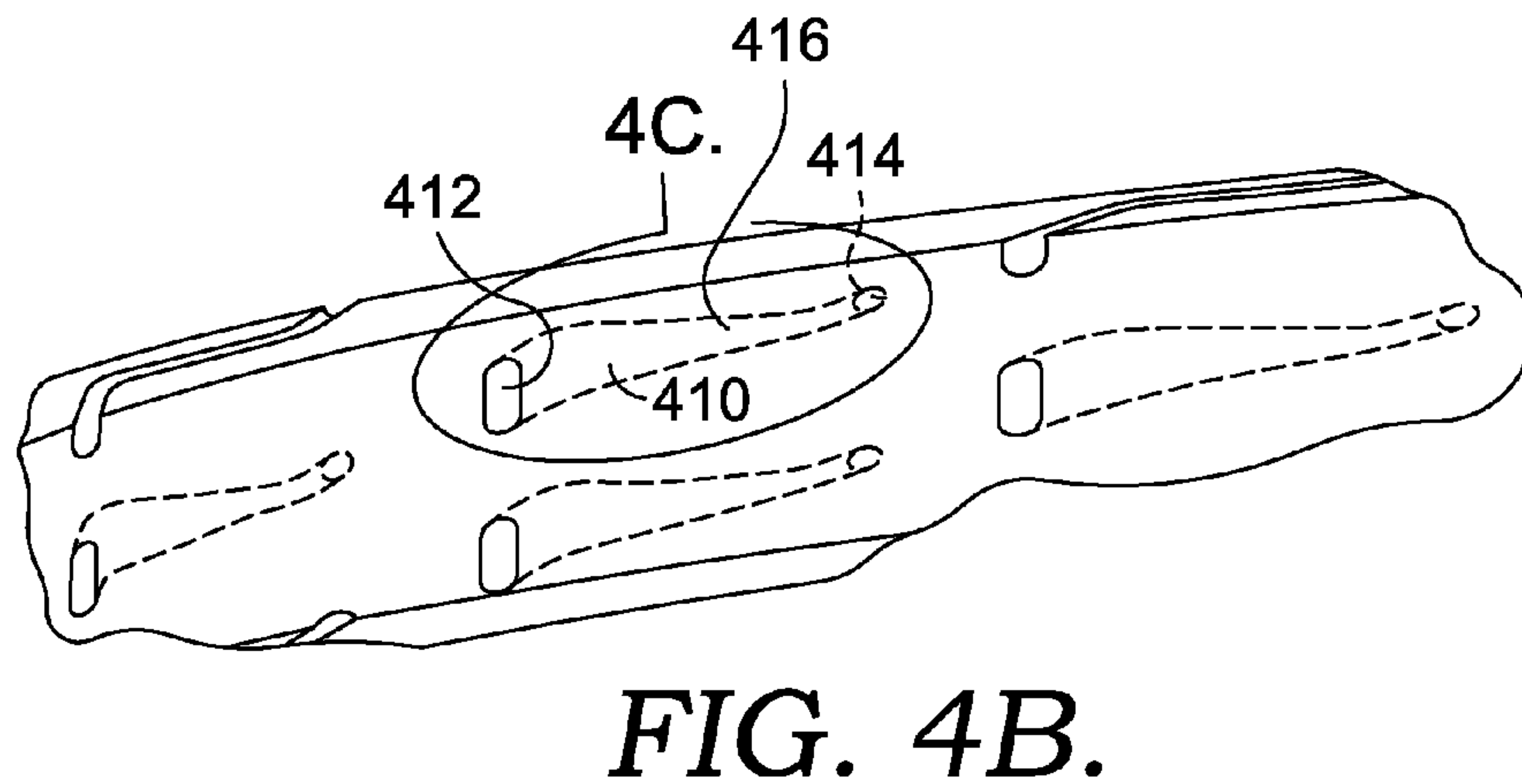
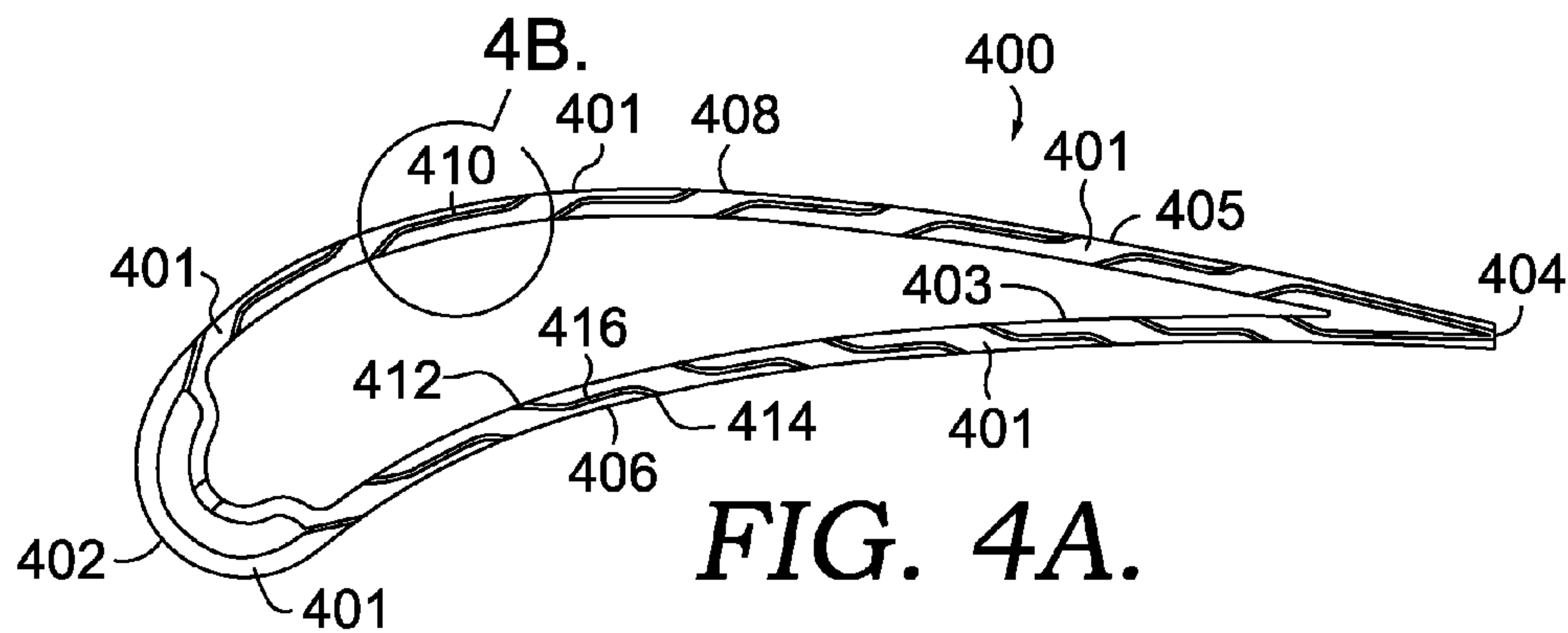


FIG. 2.
PRIOR ART





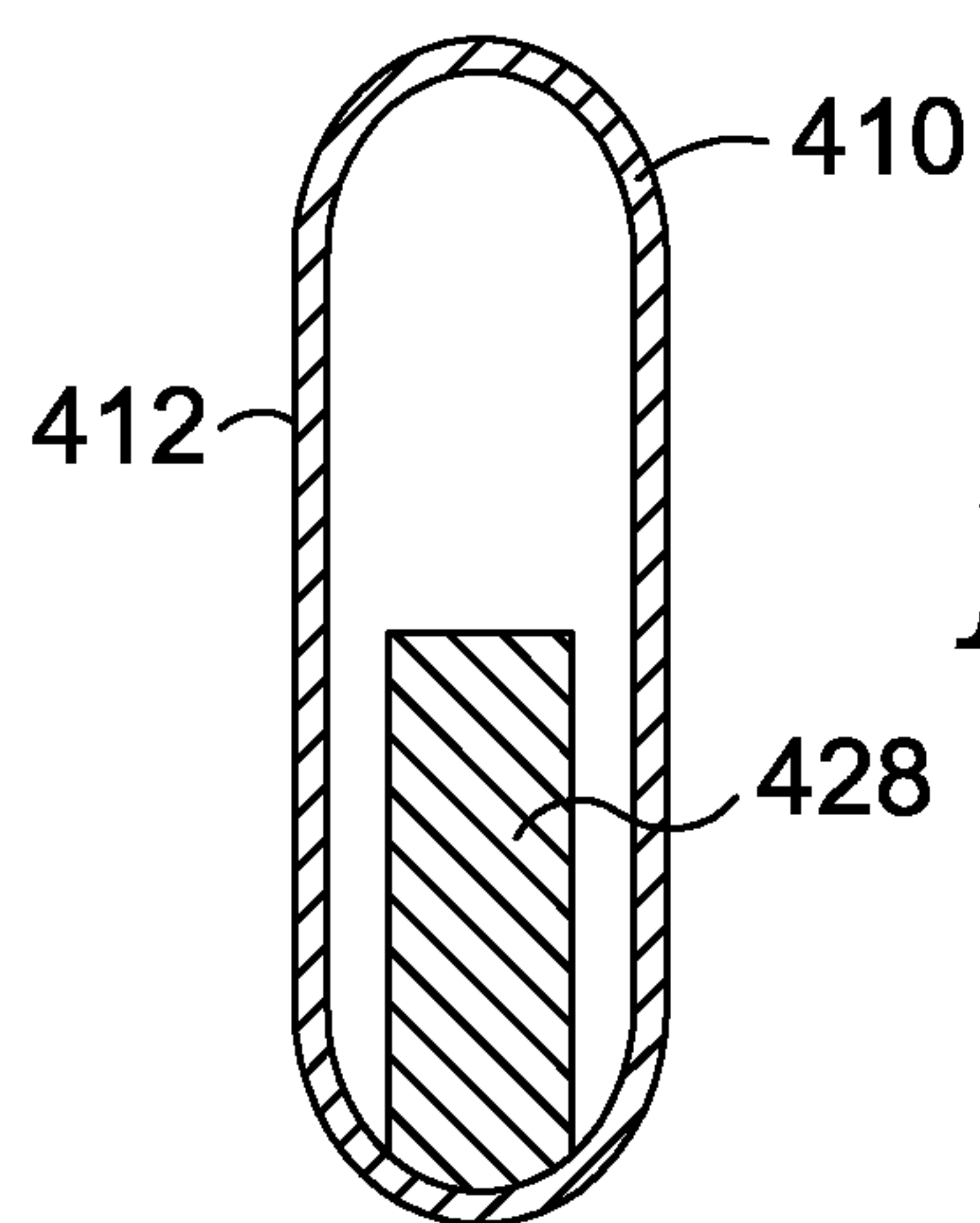


FIG. 4D.

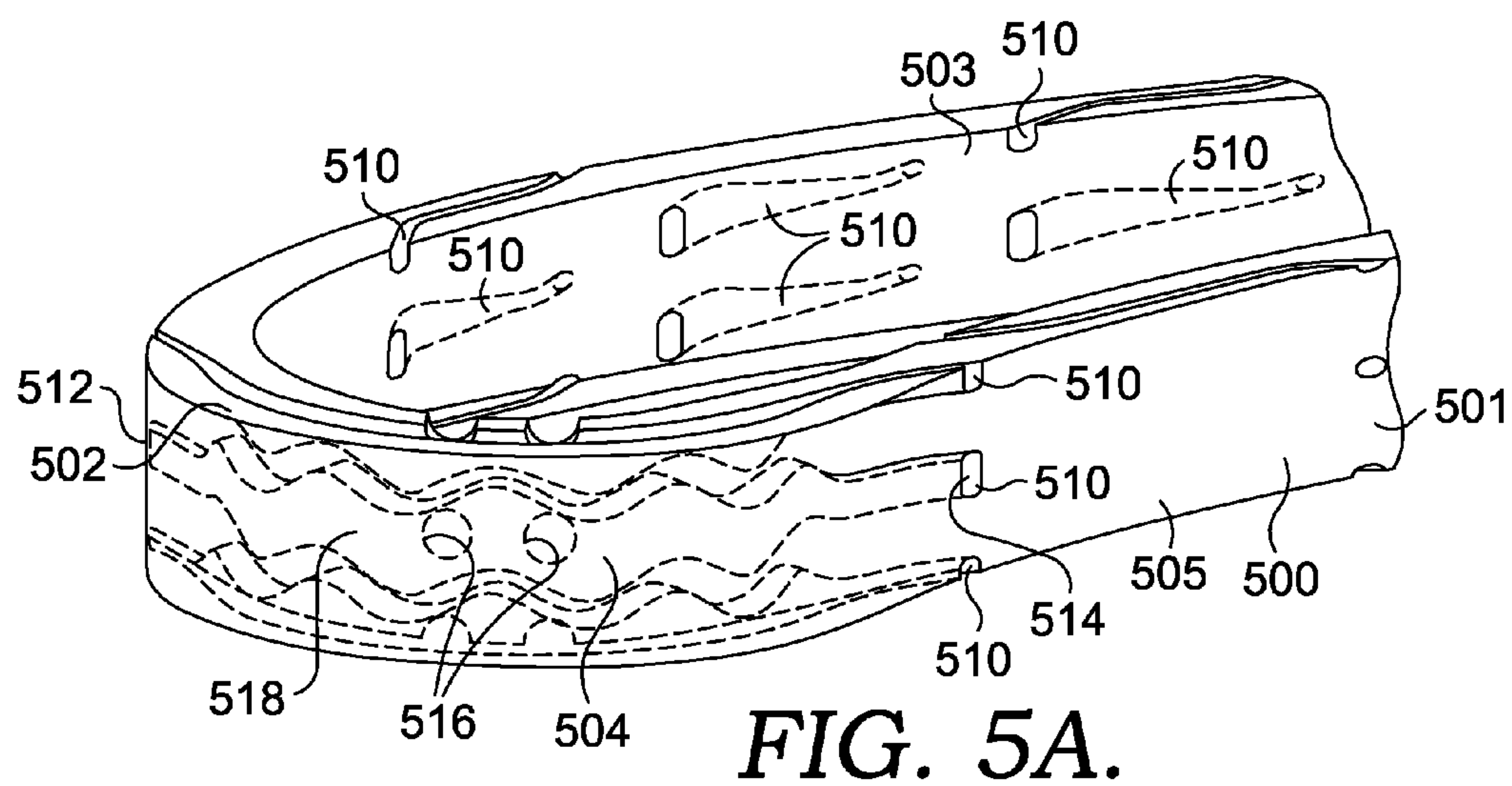


FIG. 5A.

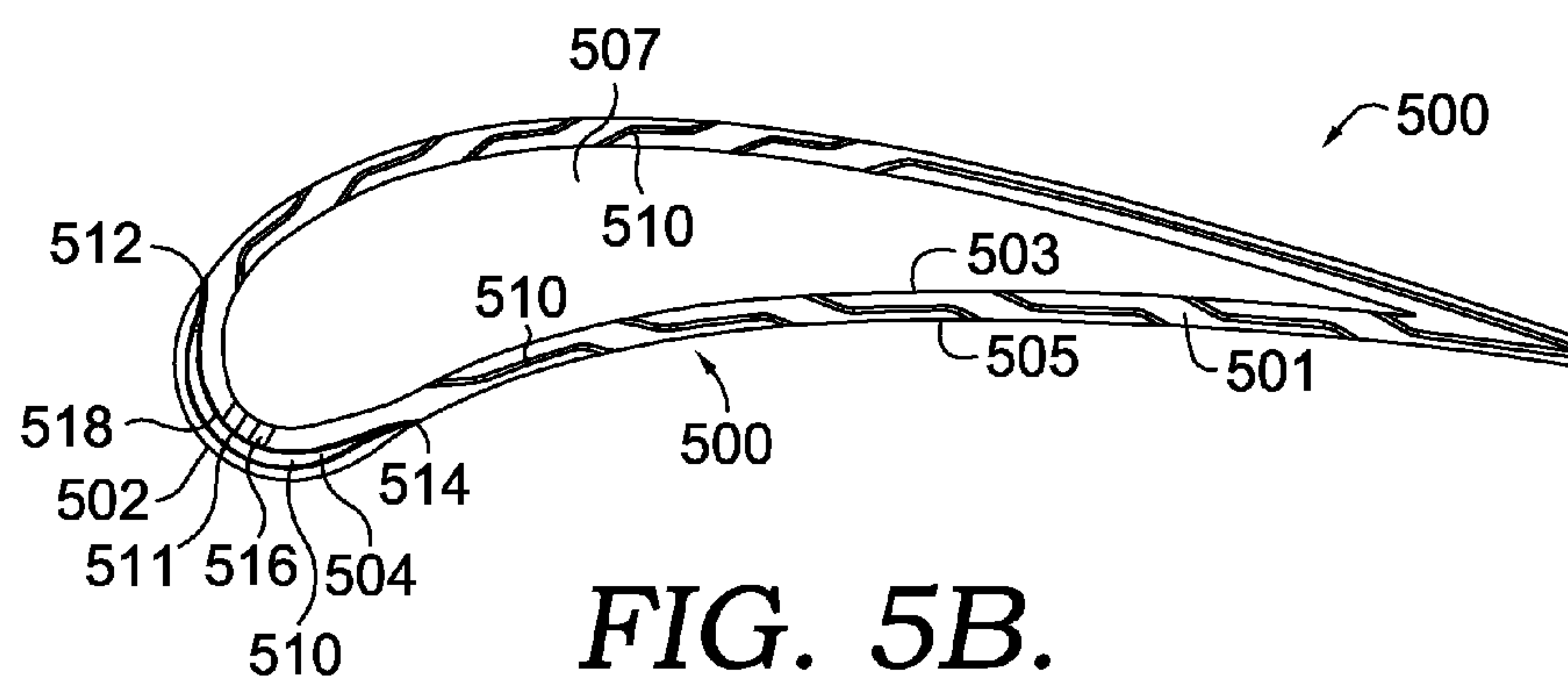


FIG. 5B.

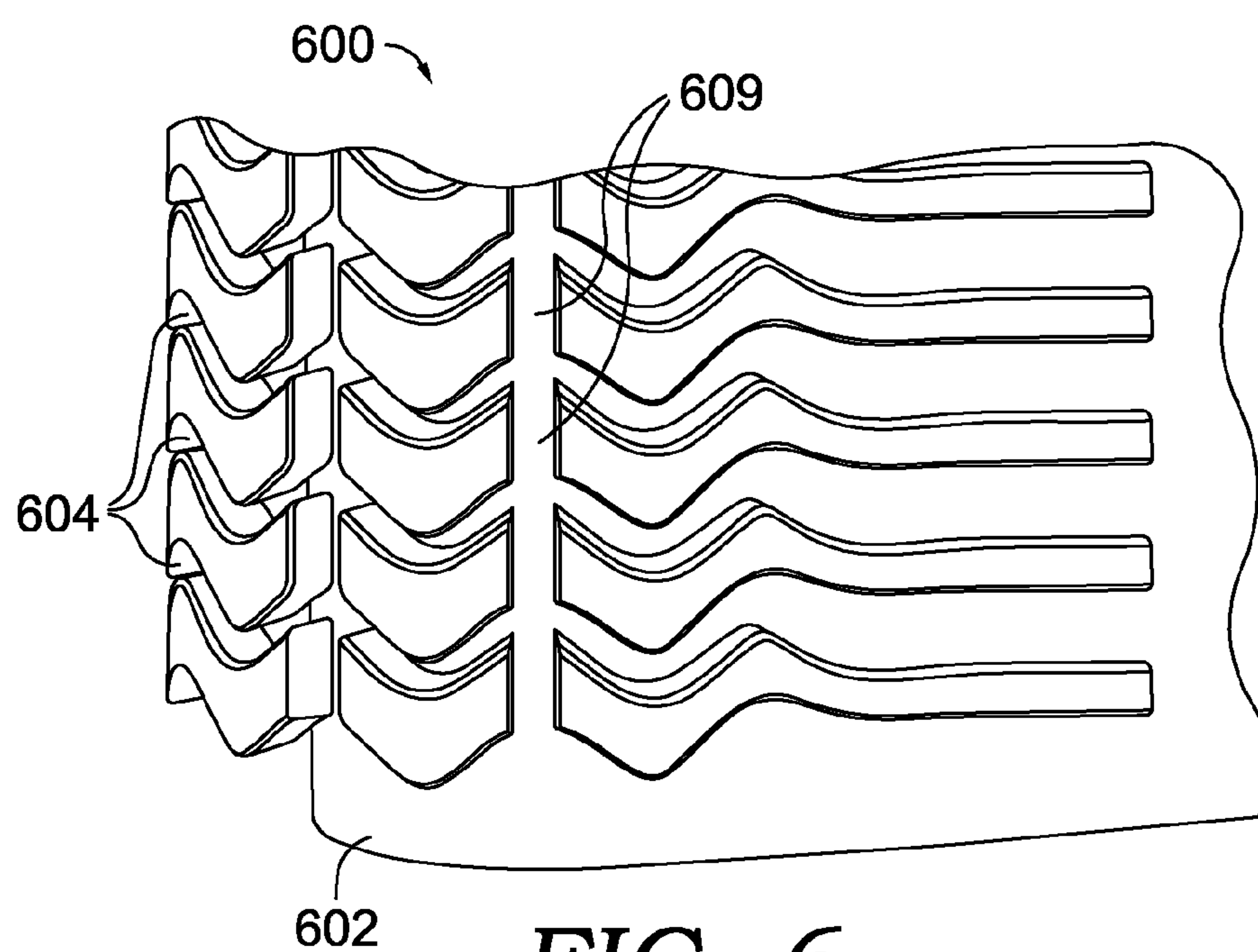


FIG. 6.

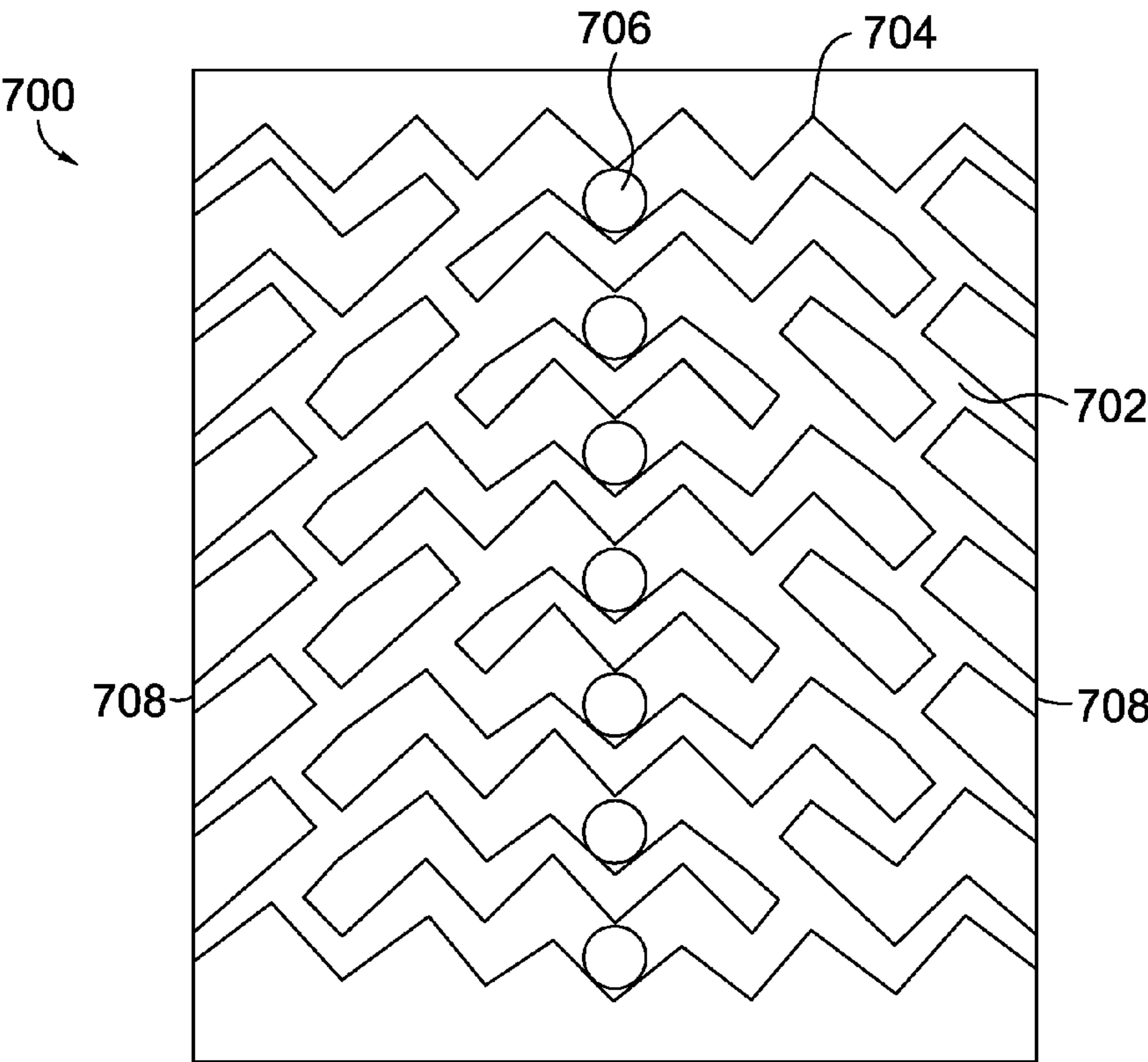


FIG. 7A.

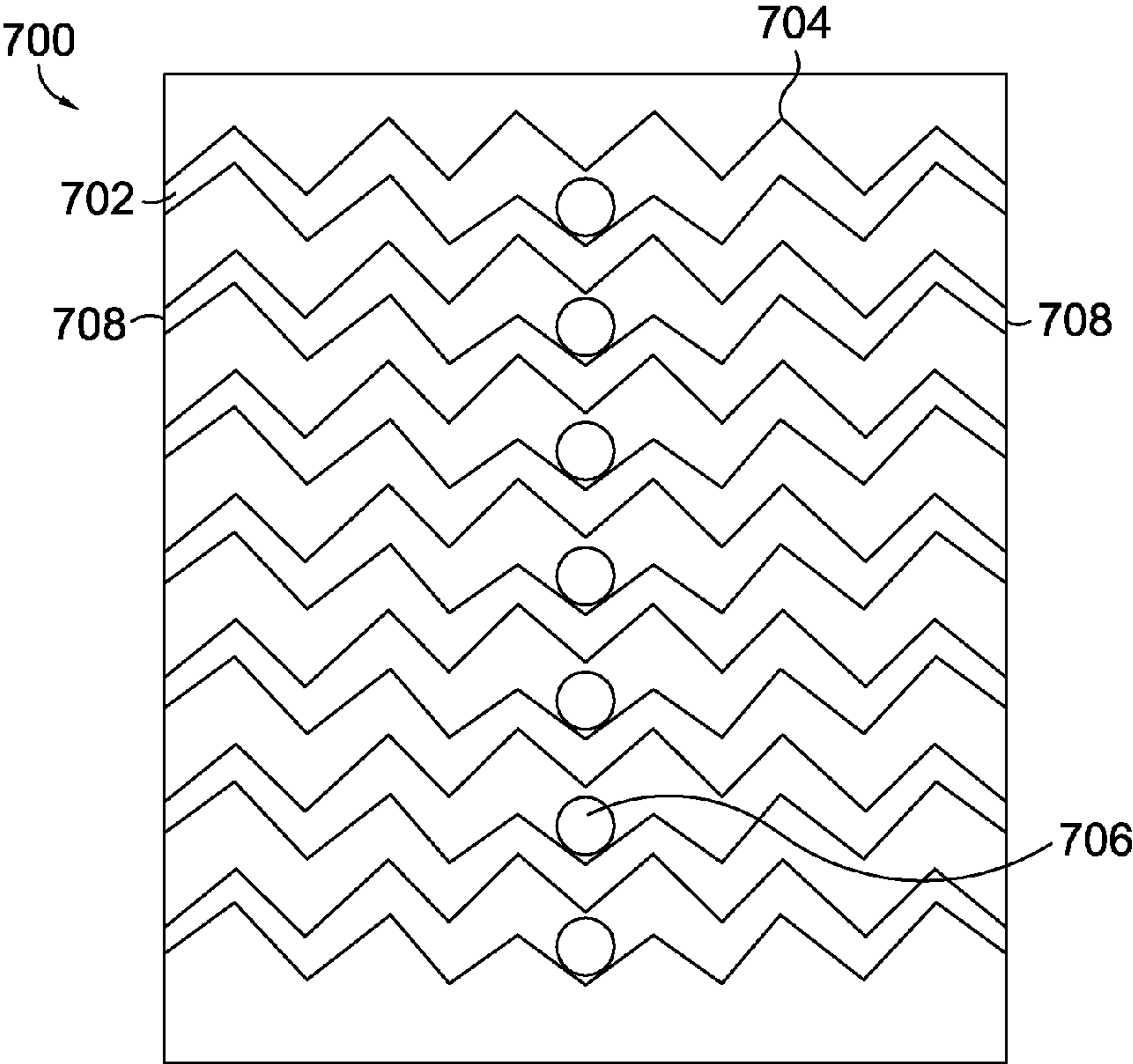


FIG. 7B.

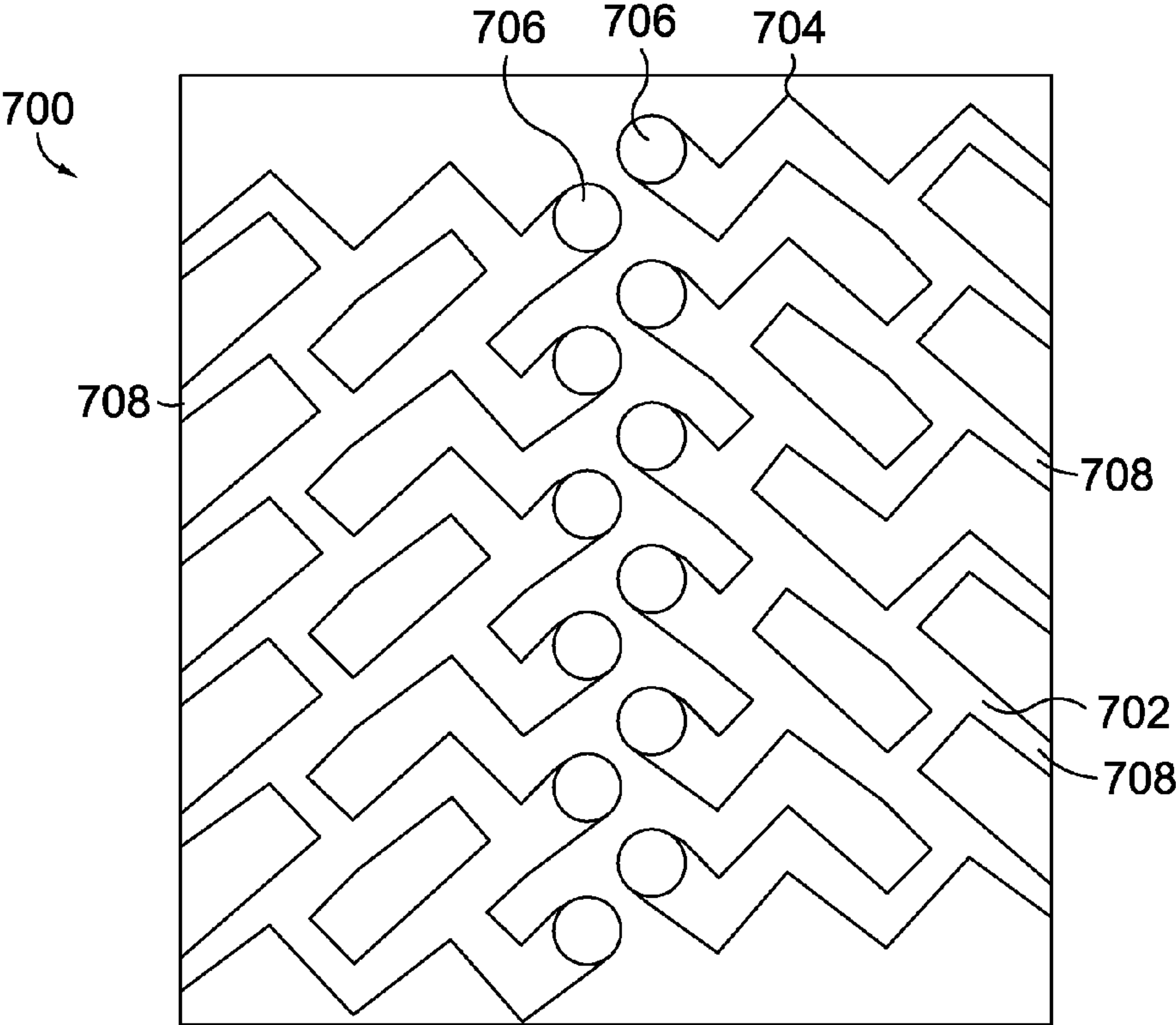


FIG. 7C.

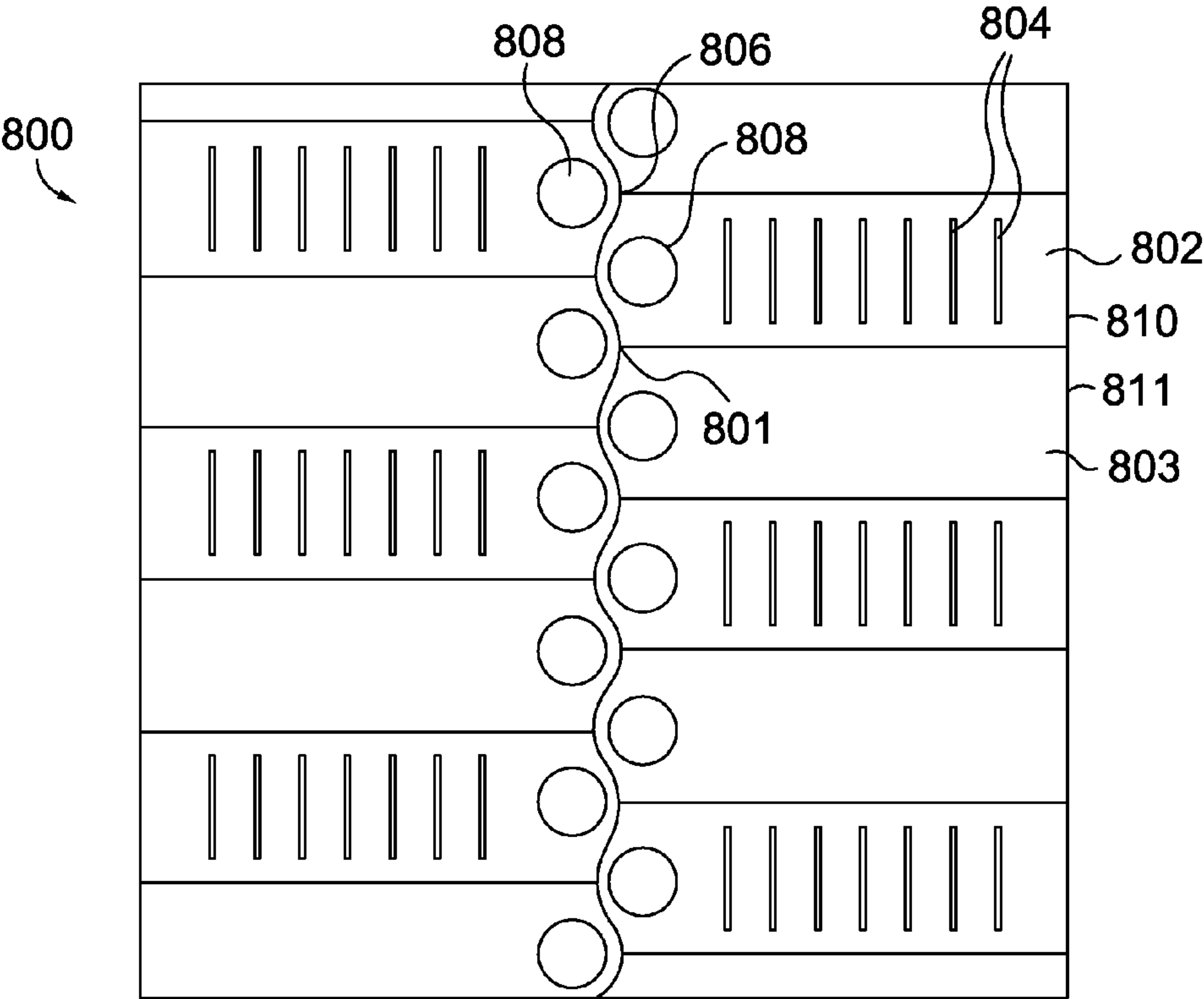


FIG. 8.

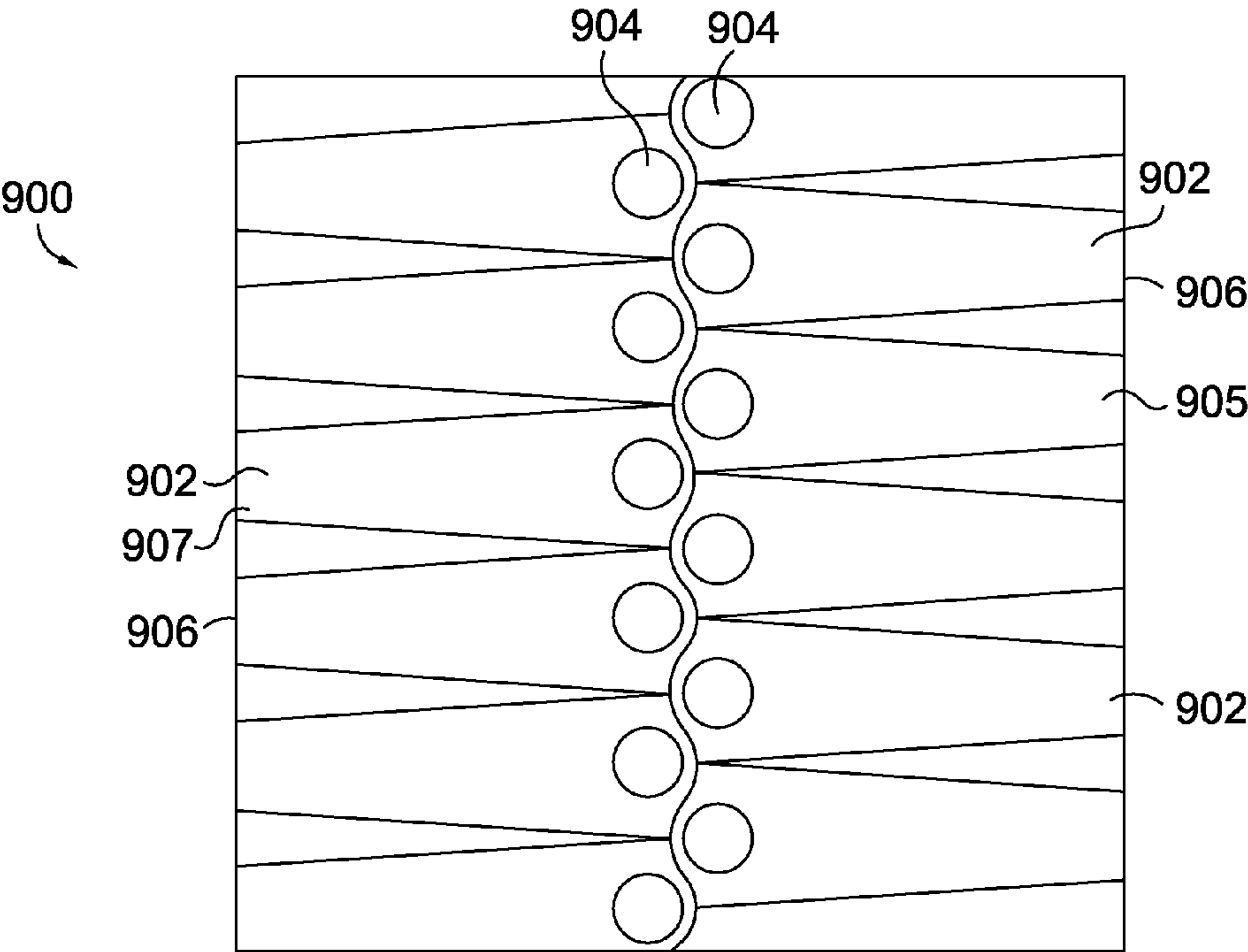


FIG. 9A.

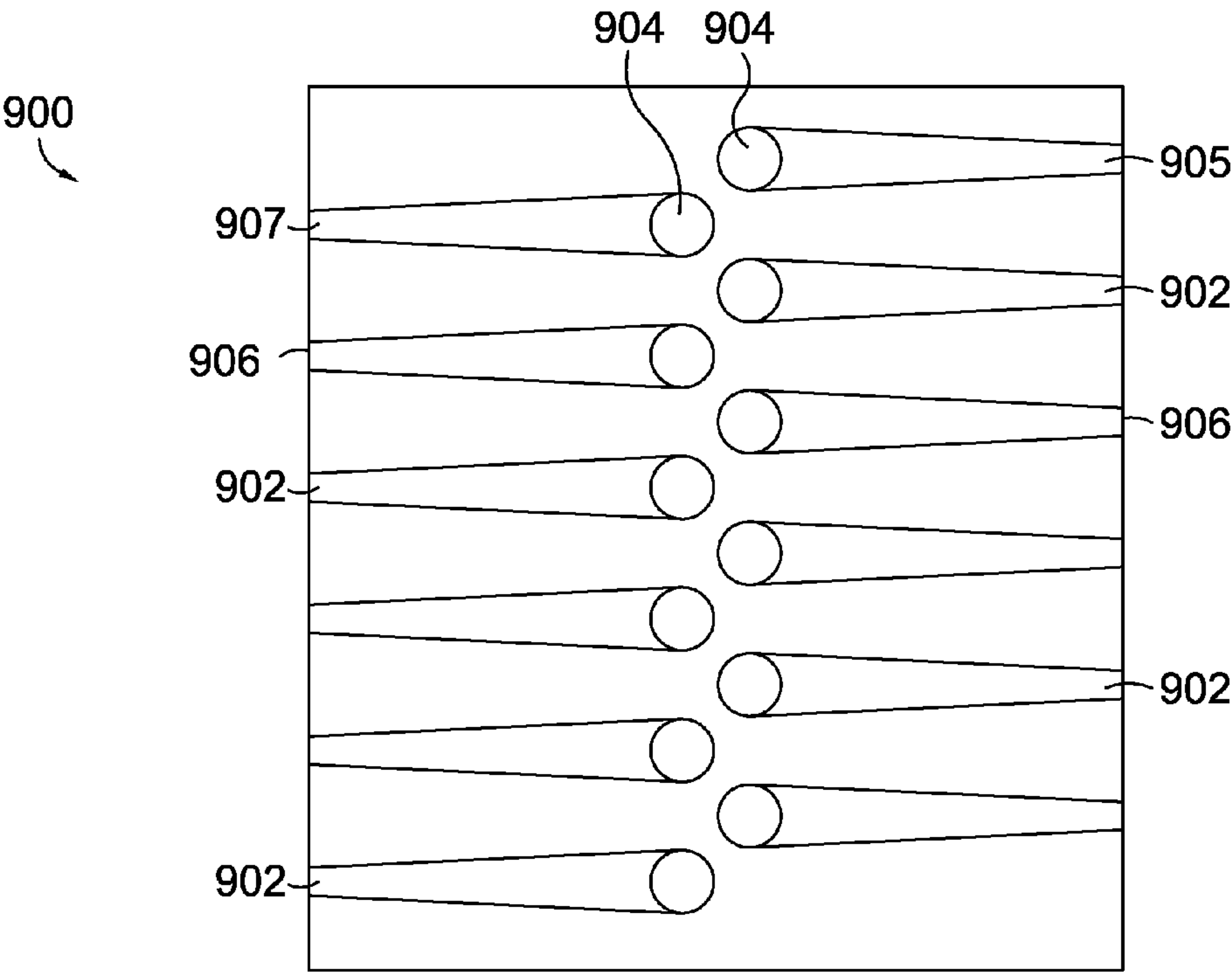


FIG. 9B.

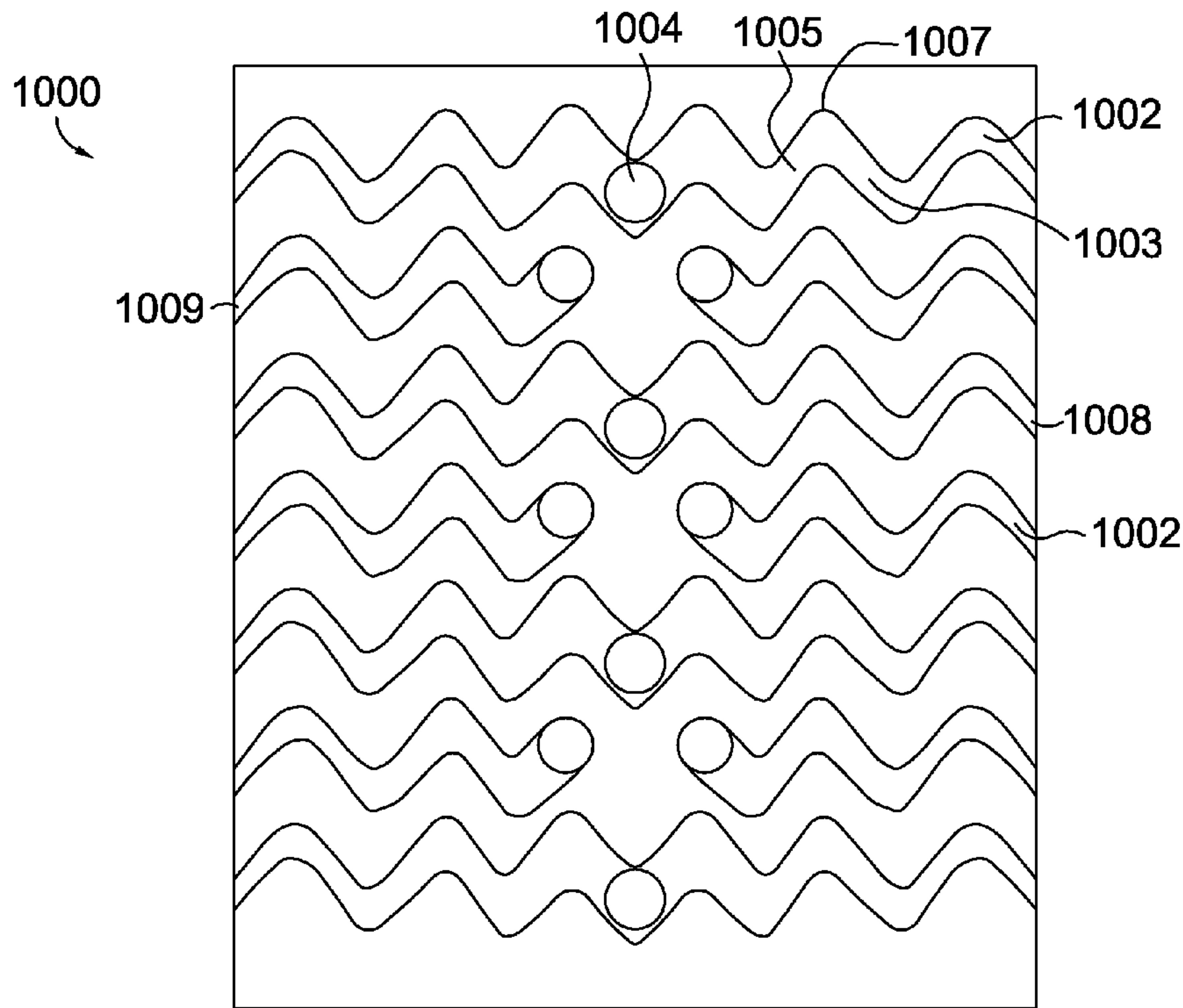


FIG. 10A.

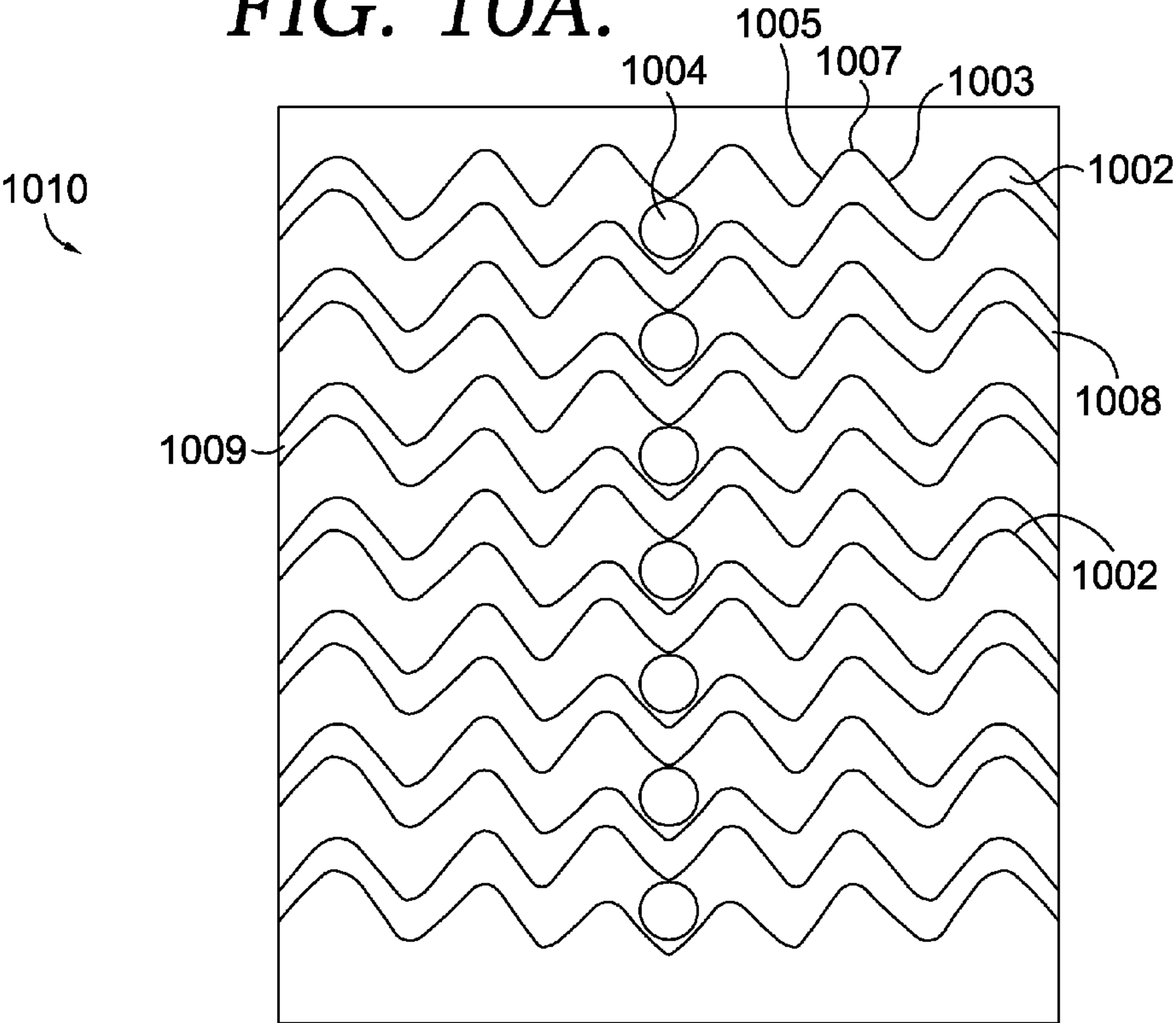


FIG. 10B

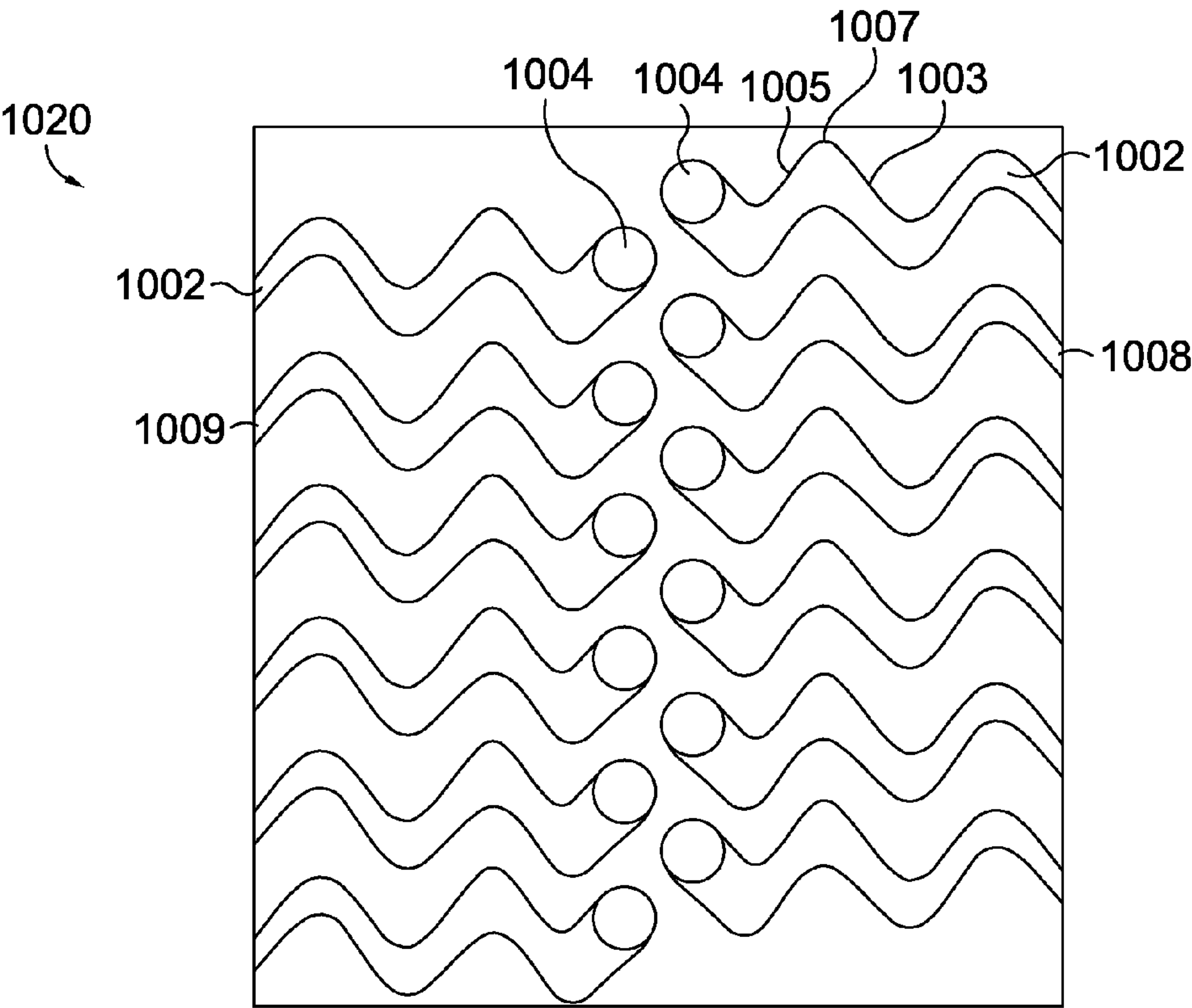


FIG. 10C.

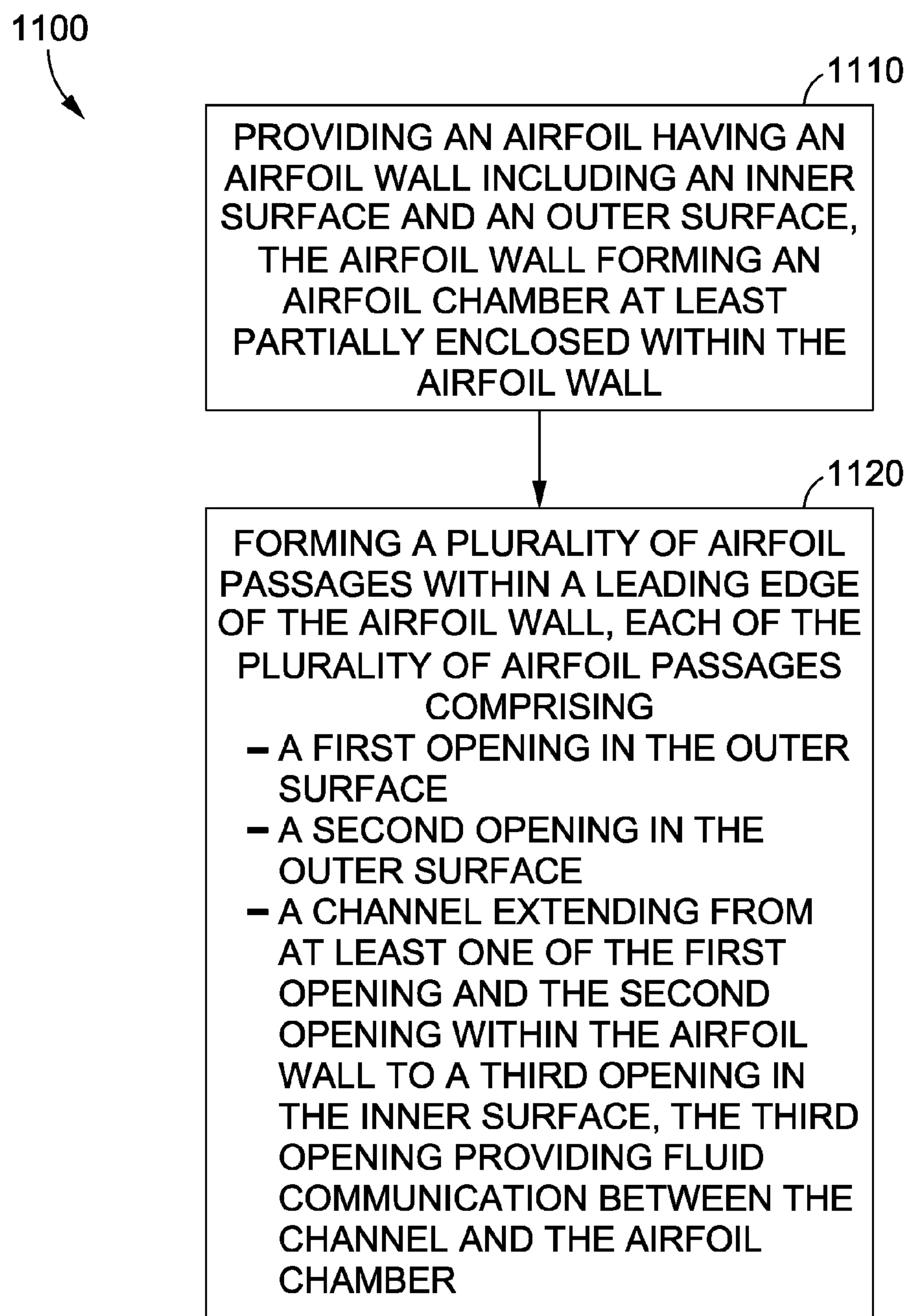


FIG. 11.

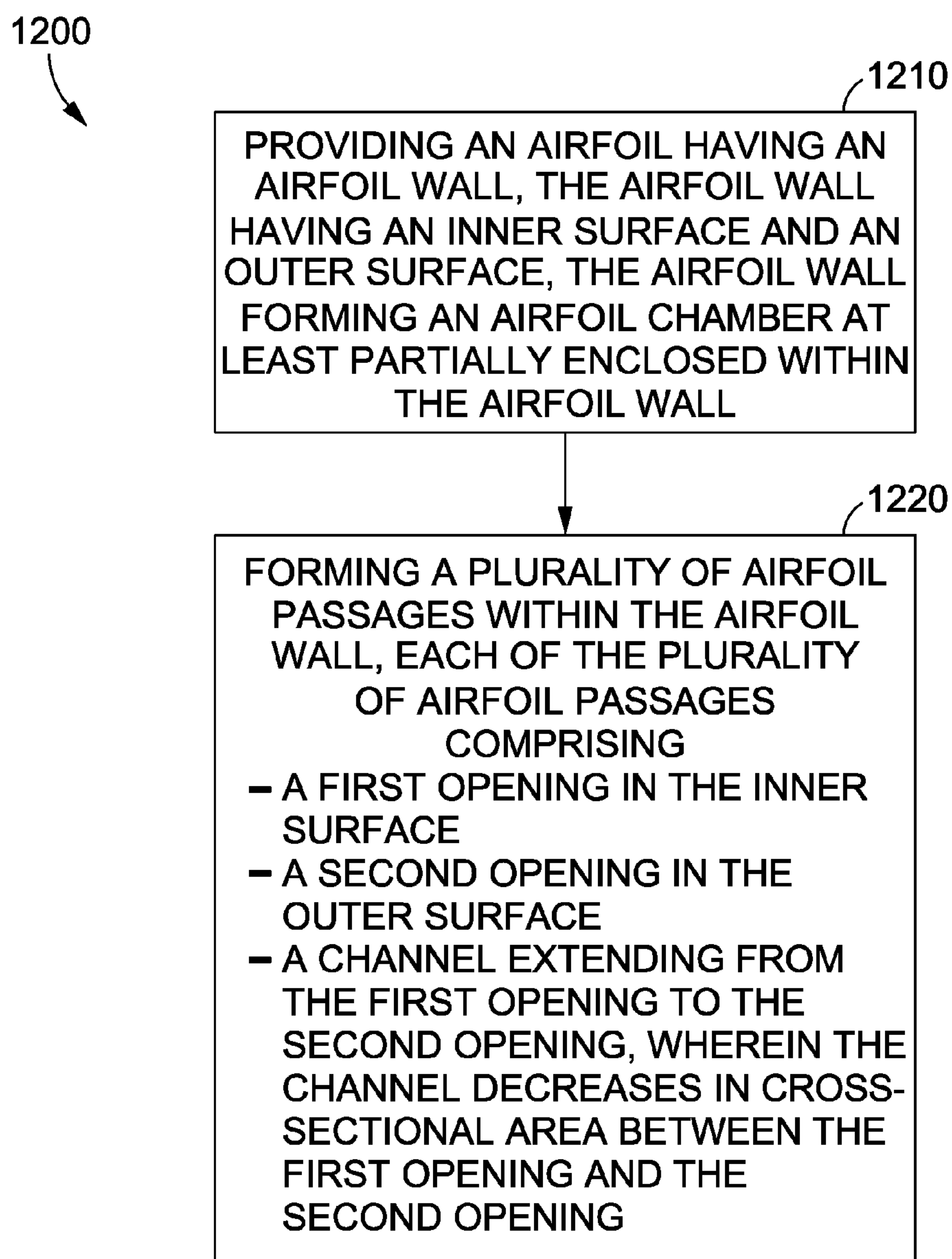
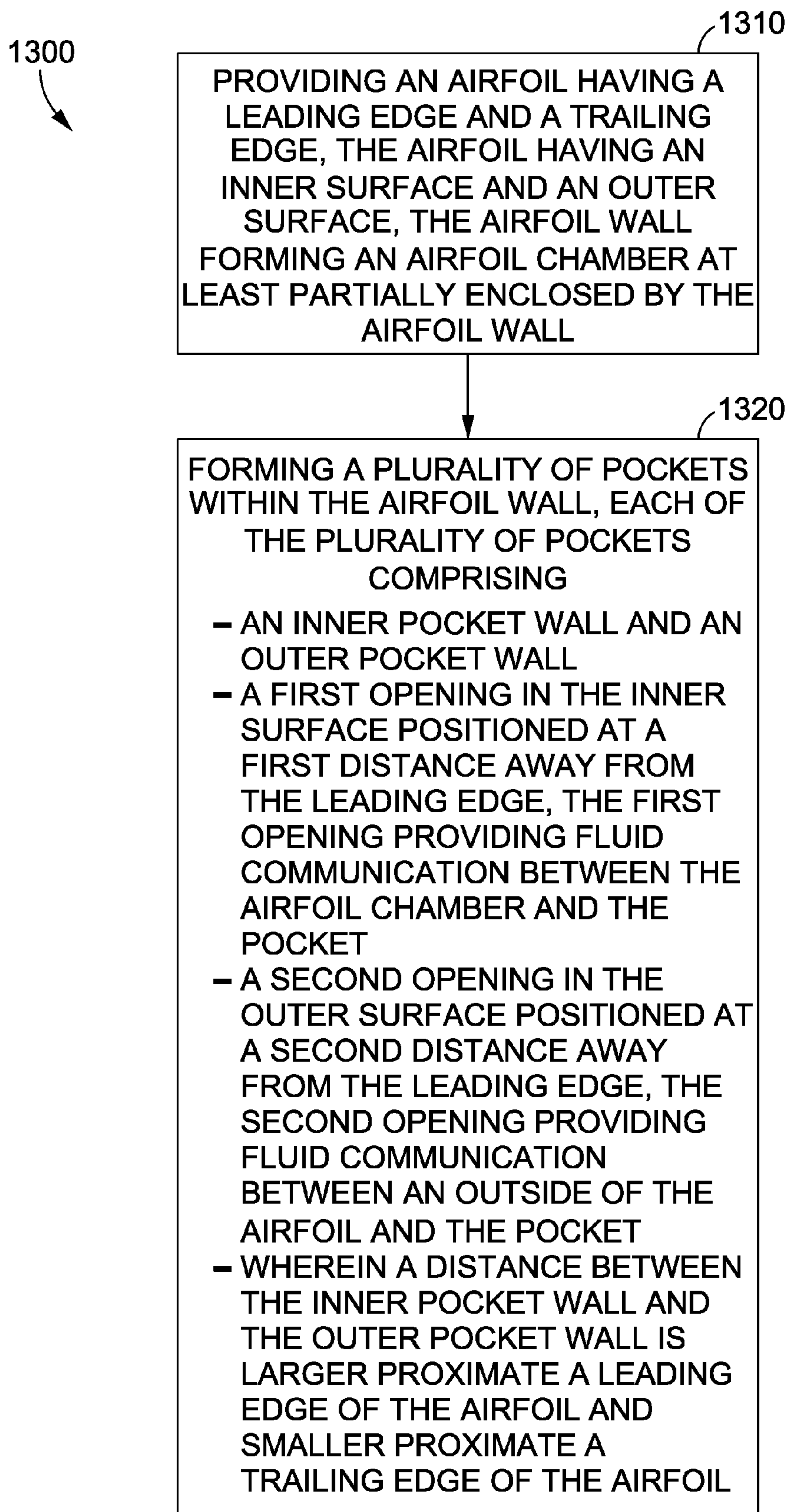


FIG. 12.

*FIG. 13.*

COOLING CHANNEL FOR AIRFOIL WITH TAPERED POCKET

CROSS-REFERENCE TO RELATED APPLICATIONS

[0001] This Application claims the benefit of U.S. Provisional Application No. 62/084,810, filed Nov. 26, 2014, and titled “GAS TURBINE AIRFOIL WITH TAPERED AIR-FLOW MICRO CIRCUITS FOR IMPROVED COOLING,” which is incorporated herein by reference in its entirety. This application is also related by subject matter to concurrently filed U.S. patent application Ser. No. (not yet assigned; Attorney Docket No. PSSF.241854), filed Nov. 24, 2015, and titled “LEADING EDGE COOLING CHANNEL FOR AIRFOIL,” and concurrently filed U.S. patent application Ser. No. (not yet assigned; Attorney Docket No. PSSF.241855), filed Nov. 24, 2015, and titled “TAPERED COOLING CHANNEL FOR AIRFOIL.” The teachings of each of these concurrently filed applications are also incorporated herein by reference in their entirety.

TECHNICAL FIELD

[0002] The present invention relates to turbine airfoils, and more particularly, to cooling circuits incorporated into turbine airfoils.

BACKGROUND OF THE INVENTION

[0003] A typical gas turbine engine is comprised of three main sections: a compressor section, a combustor section, and a turbine section. When in a standard operating cycle, the compressor section is used to pressurize air supplied to the combustor section. In the combustor section, a fuel is mixed with the pressurized air from the compressor section and is ignited in order to generate high temperature and high velocity combustion gases. These combustion gases then flow into a multiple stage turbine, where the high temperature gas flows through alternating rows of rotating and stationary gas turbine airfoils. The rows of stationary vanes are typically used to redirect the flow of combustion gases onto a subsequent stage of rotating blades. The turbine section is coupled to the compressor section along a common axial shaft, such that the turbine section drives the compressor section.

[0004] The air and hot combustion gases are directed through a turbine section by turbine blades and vanes. These blades and vanes are subject to extremely high operating temperatures, often exceeding the material capability from which the blades and vanes are made. Extreme temperatures can also cause thermal growth in the components, thermal stresses, and can lead to durability shortfall. In order to lower the effective operating temperature, the blades and vanes are cooled, often with air or steam. However, the cooling must occur in an effective way so as to use the cooling fluid efficiently. As a result, an improved cooling design for airfoils in gas turbines that addresses these issues, among others, is needed.

BRIEF SUMMARY OF THE INVENTION

[0005] In brief, and at a high level, the subject matter of this application relates generally to cooling passages, channels, and chambers incorporated into gas turbine airfoils. A gas turbine airfoil is comprised of an airfoil wall that includes an inner surface and an outer surface, and that forms an airfoil chamber that is at least partially enclosed by the airfoil wall.

Embodiments provide for airfoil passages and pockets that are formed in various locations, directions, and configurations in the airfoil wall for improved cooling of the airfoil. The airfoil passages allow for cooling fluid or air to pass through the airfoil wall and airfoil chamber, cooling the airfoil during operation of the gas turbine.

[0006] In a first embodiment of the invention, an airfoil for a gas turbine having a leading edge and a trailing edge is provided. The airfoil comprises an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall, and a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising an inner pocket wall and an outer pocket wall, at least one first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket, and at least one second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the pocket. A distance between the inner pocket wall and the outer pocket wall is larger proximate the leading edge and smaller proximate the trailing edge.

[0007] In another embodiment of the present invention, a gas turbine assembly is provided. The assembly comprises an airfoil having a leading edge and a trailing edge, the airfoil comprising an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall, and a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising an inner pocket wall and an outer pocket wall, at least one first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket, and at least one second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the respective pocket. A distance between the inner pocket wall and the outer pocket wall is larger proximate the leading edge and smaller proximate the trailing edge, and a ratio of cross-sectional area of the inner pocket wall to the outer pocket wall is between 1.1:1 and 10:1.

[0008] In yet another embodiment of the present invention, a method of manufacturing airfoils is provided. The method comprises providing an airfoil having a leading edge and a trailing edge, the airfoil having an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall, and forming a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising an inner pocket wall and an outer pocket wall, a first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket, and a second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the pocket. A distance between the inner pocket wall and the outer pocket wall is larger proximate a leading edge of the airfoil and smaller proximate a trailing edge of the airfoil.

[0009] The cooling circuits, channels, passages, and/or micro-circuits described in this disclosure are discussed frequently in the context of gas turbine airfoils, but may be used

in any type of airfoil structure. Additionally, cooling fluid, gas, air, and/or airflow may be used interchangeably in this disclosure, and refer to any cooling medium that can be sent through an airfoil to provide heat transfer and cooling of the airfoil.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWING

[0010] The present invention is described in detail below with reference to the attached drawing figures, wherein:

[0011] FIG. 1A is a perspective view of a prior art gas turbine airfoil;

[0012] FIG. 1B is a perspective view of a prior art gas turbine vane;

[0013] FIG. 2 is a cross-sectional view of the airfoil shown in FIG. 1A;

[0014] FIG. 3A is an angled, perspective, cross-sectional view of an airfoil with cooling channels, in accordance with an embodiment of the present invention;

[0015] FIG. 3B is a cross-sectional view of the airfoil shown in FIG. 3A, in accordance with an embodiment of the present invention;

[0016] FIG. 3C is a partial, cross-sectional, perspective view of a cooling pocket of the airfoil shown in FIGS. 3A and 3B, in accordance with an embodiment of the present invention;

[0017] FIG. 4A is a cross-sectional view of an airfoil with a first configuration of cooling channels, in accordance with an embodiment of the present invention;

[0018] FIG. 4B is a partial, perspective, cross-sectional view of the airfoil shown in FIG. 4A, in accordance with an embodiment of the present invention;

[0019] FIG. 4C is a perspective view of a radially tapering airfoil passage which can be formed into an airfoil wall, in accordance with an embodiment of the present invention;

[0020] FIG. 4D is a cross-sectional view of the airfoil passage shown in FIG. 4C incorporated into an airfoil wall and including a flow turbulator, in accordance with an embodiment of the present invention;

[0021] FIG. 5A is a perspective view of an airfoil having multiple cooling channels, in accordance with an embodiment of the present invention;

[0022] FIG. 5B is a cross-sectional, elevation view of the airfoil shown in FIG. 5A, in accordance with an embodiment of the present invention;

[0023] FIG. 6 is a partial, angled, perspective view of cooling channels incorporated into a leading edge of an airfoil, in accordance with an embodiment of the present invention;

[0024] FIGS. 7A, 7B, and 7C are cut views of various cooling channel designs which can be incorporated into an airfoil, in accordance with embodiments of the present invention;

[0025] FIG. 8 is a cut view of an alternate cooling channel design, in accordance with an embodiment of the present invention;

[0026] FIGS. 9A and 9B are cut views of alternate cooling channel designs, in accordance with embodiments of the present invention;

[0027] FIGS. 10A, 10B, and 10C are cut views of alternate cooling channel designs, in accordance with embodiments of the present invention;

[0028] FIG. 11 is a block diagram of an exemplary method of manufacturing gas turbine airfoils, in accordance with an embodiment of the present invention;

[0029] FIG. 12 is a block diagram of an exemplary method of manufacturing gas turbine airfoils, in accordance with an embodiment of the present invention; and

[0030] FIG. 13 is a block diagram of an exemplary method of manufacturing gas turbine airfoils, in accordance with an embodiment of the present invention.

DETAILED DESCRIPTION OF THE INVENTION

[0031] At a high level, the subject matter of this application generally relates to an airfoil for a gas turbine that includes cooling circuits integrated in various configurations. The airfoil may generally include an airfoil wall with an inner surface and an outer surface that at least partially encloses an airfoil chamber. Cooling circuits may be formed in various locations in the airfoil wall, to provide enhanced heat transfer from the airfoil when the gas turbine is in operation and cooling fluid or gas is passing through the cooling circuits. For turbine hardware operating in harsh environments, the use of this airfoil cooling technology is fully contemplated to be adapted to additional components such as outer and inner diameter platforms, blade outer or inner air shields, or alternative high temperature turbine components.

[0032] Referring now to FIG. 1A, a gas turbine blade 100 is provided. The turbine blade 100 comprises a bottom portion commonly referred to as a root 102, which may be coupled to a rotor disk (not shown). It is understood that the root may be completely integrated into the rotor disk, such that the root does not extend into the flow path. Extending in an upward radial, typically perpendicular to the rotor central axis, direction from the root 102 is the neck 103. The neck 103 may primarily be used as a transitional piece between the root 102 and the gas turbine airfoil 104.

[0033] The gas turbine airfoil 104 is comprised of four distinct portions. The first portion of the airfoil 104 that comes into contact with pressurized gas flow is referred to as the leading edge 106, which is opposed by the last portion of the airfoil to come in contact with the gas flow, defined as the trailing edge 108. The leading edge 106 faces the turbine compressor section (not shown), or turbine inlet, along the rotor center axis. This direction is referred to as the axial direction. When pressurized airflow impedes upon the leading edge 106, the airflow splits into two separate streams of air with different relative pressures. Connecting the leading edge 106 and the trailing edge 108 are two radially extending walls, which are defined based on the relative pressures impeding on the walls. The concave surface seen in FIG. 1A is defined to be a pressure side wall 110. The concave geometry of this surface generates a higher local pressure along the length of the pressure side wall 110. Opposing the pressure side wall 110 is a suction side wall 112. The suction side wall 112 has a convex geometry, which generates a lower local pressure along the length of the suction side wall 112.

[0034] The pressure differential created between the pressure side wall 110 and the suction side wall 112 creates an upward lifting force along the cross-section of the gas turbine airfoil 104. The cross-section of the gas turbine airfoil 104 can be seen in greater detail in FIG. 2. This lifting force actuates the rotational motion of the rotor disk. The rotor disk may be coupled to a compressor and a generator via a shaft (not shown) for the purposes of generating electricity. The uppermost portion of FIG. 1A shows a tip shroud 114 containing a first surface 116 that is populated with knife edges 118 that extend radially outward from the first surface 116. Located between the knife edges 118 are recessed pockets 120.

[0035] A vane assembly 150 of the prior art is shown in FIG. 1B, and comprises an inner platform 151, inner rail 152, outer platform 153, and vane airfoils 154 extending between inner platform 151 and outer platform 153. While the inner rail 152 serves as a means to seal the rim cavity region from leakage of the cooling air into the hot gas path instead of passing to the designated vanes, inner rail 152 also stiffens inner platform 151. Inner rail 152 may be located proximate the plenum of cooling air and therefore operates at approximately the temperature of the cooling air.

[0036] FIG. 2 is a cross-sectional view of a prior art cooling design for a gas turbine airfoil. FIG. 2 is cross-sectional for the purposes of showing cooling passages 202 and 203. Gas turbine airfoils may operate in an environment where temperatures exceed the melting point of the materials used to construct the airfoil. Therefore, cooling passages 202 and 203 are provided as a way to decrease the temperature of the airfoil during operation by flowing cooling air through the cooling passages of the airfoil.

[0037] Traditionally, air cooled turbine airfoils are produced by a machining process or an investment casting process by forming a wax body of the turbine airfoil, providing an outer shell about the wax part, and then melting the wax to leave a mold for the liquid metal. Then, liquid metal is poured into the mold to fill the void left by the wax. Often-times the wax also contains a ceramic core to establish cooling channels within the metal turbine airfoils. Once the liquid metal cools and solidifies, the shell is removed and the ceramic core is chemically leached out of the now solid metal turbine airfoil, resulting in a hollow turbine airfoil. These traditional casting methods have limits as to the geometry that can be cast. New developments in additive manufacturing have occurred which can expand the capabilities beyond traditional investment casting techniques.

[0038] The turbine airfoils of FIGS. 1A, 1B, and 2 are known to be manufactured using standard metallurgy techniques, such as investment casting. However, the geometries that can be created using traditional manufacturing technique are limited. Internal geometrical shapes, as well as small geometrical intricacies, are generally not suitable for die casting. Advances in the field of additive manufacturing, have been adopted for the manufacturing of intricacies that were previously unattainable. The embodiments of the present invention may be created using an additive manufacturing process. An example of an additive manufacturing process is selective laser melting, known more commonly in the manufacturing field as SLM. Although SLM is widely considered a common additive manufacturing process, the embodiments described herein can be manufactured with any additive manufacturing process, such as selective laser sintering (SLS) or direct metal laser sintering (DMLS) or an alternative additive manufacturing method. The SLM processes described herein are intended to be non-limiting and exemplary.

[0039] FIGS. 3A and 3B are cross-sectional perspective views of an exemplary gas turbine airfoil 300 incorporating various cooling channels, in accordance with an embodiment of the present invention. The airfoil 300 includes an airfoil wall 301 having an inner surface 303 and an outer surface 305. The airfoil wall 301 at least partially encloses an airfoil chamber 307 within the airfoil wall 301. The airfoil wall 301 as a whole comprises a leading edge 302, a trailing edge 304, a pressure side wall 306, and a suction side wall 308. Positioned within the pressure side wall 306 are pockets 310 and

312. Pockets 314 and 316 are positioned within the suction side wall 308. These pockets 310, 312, 314, and 316 have been introduced into the airfoil wall 301 of the gas turbine airfoil 300 for the purpose of increasing active cooling within the airfoil 300 by allowing cooling fluid or gas to pass through interior portions of the airfoil wall 301 to carry heat away from the airfoil 300 during operation of an associated gas turbine to which the airfoil 300 is coupled.

[0040] Additionally, the pocket sections 310, 312, 314, and 316 (which are shown by the spaces within the airfoil wall 301) may be manufactured using an additive manufacturing process, as previously discussed. As shown in FIGS. 3A and 3B, pockets 310, 312, 314, and 316 each extend within the airfoil wall 301, and each include a first opening 318, which may be one of a plurality of first openings 318, referred to hereinafter as the first opening 318 for simplicity but intended to be non-limiting, (which may be a cooling fluid inlet) on the inner surface 303, and a second opening 320, which may be one of a plurality of second openings 320, referred to hereinafter as the second opening 320 for simplicity but intended to be non-limiting (which may be a cooling fluid outlet) on the outer surface 305. These openings 318, 320 are provided and paired for each of the pockets 310, 312, 314, and 316. The first opening 318 of each of the pockets 310, 312, 314 and 316 provides fluid communication between the airfoil chamber 307 and the respective pocket 310, 312, 314 or 316, and the second opening 320 provides fluid communication between the respective pockets 310, 312, 314 or 316 and an outside environment of the airfoil 300. These openings 318, 320 feed and exhaust the interior pockets 310, 312, 314, and 316 of the airfoil shown in FIGS. 3A-3C.

[0041] Included within each of the pockets 310, 312, 314, and 316 of the airfoil wall 301 are a plurality of pedestals 322, which extend between an inner pocket wall 324 and an outer pocket wall 326 of each of the pocket 310, 312, 314, and 316. The pockets 310, 312, 314, and 316 may each include one or more flow turbulators (not shown), which may be extruded portions of the pocket 310, 312, 314, or 316 that promote turbulent mixing of cooling fluid or gas, to provide further sidewall cooling. These can be implemented or included as various different structures or extrusions, simply to provide mixing of cooling fluid traveling between the respective first opening 318 and respective second opening 320 within the pockets 310, 312, 314, and 316. Turbulation may alternatively be achieved by manufacturing pockets having a rough surface. The topography of a surface with roughness is complex and there is no single definitive measure of roughness. A widely used basic perimeter is "equivalent roughness" (Ra), defined as the arithmetic average of the absolute values of the measured profile height deviations of the surface from the surface profile centerline within a given sampling length. Typical values of Ra for turbomachinery components are 125 micro-inches for material as cast and 25 micro-inches for polished components. In the disclosed embodiments, the pocket heat transfer coefficient may be additionally modified by tailoring the surface roughness to achieve an equivalent roughness measured value of at least 400 Ra.

[0042] The pockets 310, 312, 314, and 316 are included in an airfoil side wall and taper in an area generally along the axial direction from the leading edge 302 to the trailing edge 304. The taper is a reduction in cross-sectional area between the first opening 318 and second opening 320 of each respective pocket 310, 312, 314, and 316. The ratio of cross-sectional area difference between the first opening 318 and the

second opening **320** of each of the pockets **310**, **312**, **314**, and **316** may vary between 1.1:1 and 10:1, in order to accelerate the flow of cooling fluid traveling between the first opening **318** and the second opening **320** within each of the respective pockets **310**, **312**, **314**, and **316**. This results in a balance between the internal heat pick-up and heat transfer coefficient. In other words, as more heat is removed from the airfoil **300** through passage of the cooling fluid or gas through the respective pockets **310**, **312**, **314**, and **316**, the cooling fluid or gas becomes hotter and able to absorb less heat from the airfoil wall **301**, and the acceleration of the cooling fluid or gas within the respective pockets **310**, **312**, **314**, and **316** allows the cooling fluid or gas to at least partially maintain the desired heat transfer coefficient through the pockets **310**, **312**, **314**, and **316**. In this embodiment, the reduction in cross-sectional area tapers in an axial direction, as the reduction in cross-sectional area occurs in the direction of cooling passage flow between the first opening **318** and second opening **320** generally along the axis of the rotor disk (not shown).

[0043] In FIGS. 3A and 3B, the distance between the inner pocket wall **324** and outer pocket wall **326** may be larger proximate the leading edge **302** of the airfoil **300** and smaller proximate the trailing edge **304** of the airfoil **300**. This internal passage differentiation may be further characterized by a ratio of pocket length (axial or radial) to airfoil wall width. The airfoil wall width is defined as the thickness between the inner surface **303** and the outer surface **305** of the airfoil **300**. The pocket length, fully enclosed within the airfoil wall **301** in a generally axial direction, to airfoil wall width may be a minimum ratio of 1:1 to a maximum ratio dependent upon an airfoil span between the leading edge **302** and the trailing edge **304** of the airfoil **300**. This minimum ratio may also be described as the pocket length to pocket width, defined as distance between the inner pocket wall **324** and the outer pocket wall **326** measured at the first opening **318**, as a minimum ratio of 3:1.

[0044] Additionally, it is contemplated herein that each of the plurality of pedestals **322** in FIGS. 3A, 3B, and most clearly shown in FIG. 3C, may have a circular, triangular, square, ovular, or rectangular cross-sectional shape, among other shapes. Further, each of the plurality of pedestals **322** may have a non-uniform or varying cross-sectional area, for the purposes of creating optimal air flow characteristics within each pocket **310**, **312**, **314**, and **316**.

[0045] Also, in FIGS. 3A and 3B, pocket sections **310**, **312**, **314** and **316** may be arrayed in a linear or non-linear pattern within the airfoil wall **301**, or rather, not aligned linearly along the airfoil wall **301**. Further, the shape of the inner pocket wall **324** and the outer pocket wall **326** may be aligned substantially parallel to the inner surface **303** of airfoil wall **301** and/or the outer surface **305** of the airfoil wall **301**. Additionally, it is contemplated that the second opening **320** may be positioned in the pressure side wall **306** or the suction side wall **308** of the airfoil **300** for each of the corresponding pockets **310**, **312**, **314**, and **316**. These pockets **310**, **312**, **314**, and **316** may be radially arrayed and fully enclosed within the airfoil wall **301**, having a pocket height in a radial direction to airfoil wall thickness at a minimum ratio of 1:1. Further, the positioning and structure of pockets **310**, **312**, **314**, and **316** may be manufactured using additive manufacturing.

[0046] FIG. 4A is a cross-sectional view of an exemplary airfoil **400**, in accordance with an embodiment of the present invention. In FIG. 4A, the airfoil **400** comprises an airfoil wall **401**, a leading edge **402**, an inner surface **403**, a trailing edge

404, an outer surface **405**, a pressure side wall **406**, and a suction side wall **408**. The airfoil **400** further includes a plurality of airfoil passages **410**, which may allow cooling of the airfoil wall **401** when cooling fluid or gas passes through the airfoil passages **410**.

[0047] In the exemplary airfoil **400**, components of which are also shown in FIGS. 4B and 4C, the airfoil passages **410** extend from the inner surface **403** to the outer surface **405** of the airfoil wall **401** at various locations. The airfoil passages **410** in this embodiment allow cooling fluid or gas to enter a respective airfoil passage **410** at a first opening **412**, which may be one of a plurality of first openings **412**, referred to hereinafter as the first opening **412** for the sake of simplicity but intended to be non-limiting, and discharge the cooling fluid or gas from a second opening **414**, which may be one of a plurality of second openings **414**, referred to hereinafter as the second opening **414** for the sake of simplicity but intended to be non-limiting. A channel **416** extends from the first opening **412** to the second opening **414** within the airfoil wall **401**.

[0048] Additionally, in FIGS. 4A and 4B, a cross-sectional area of the channel **416** changes between the first opening **412** and the second opening **414**. The airfoil passage **410** in FIGS. 4A-4C includes a cross-sectional area change between the first opening **412** and the second opening **414** that is approximately four to one; however, it is contemplated that the cross-sectional area difference may vary from 1.1:1 to 10:1 between the first and the second opening **412**, **414**, or have another relative difference. The airfoil passage **410** in this airfoil **400** is generally described as tapering in a radial direction, as the reduction in area between the first opening **412** and the second opening **414** occurs in the direction of cooling fluid flow along the radius of the rotor disk (not shown).

[0049] FIG. 4C illustrates an enlarged perspective view of an airfoil passage **410** having the first opening **412** with a first cross-sectional area and the second opening **414** with a second cross-sectional area that is smaller than the first cross-sectional area. Additionally, the channel **416** further comprises a first section **418** having the first cross-sectional area along its axial length, a second section **420** having the second cross-sectional area along its axial length, and a transitional section **422** having a cross-sectional area that tapers between the first cross-sectional area and the second cross-sectional area of the respective first and second sections **418**, **420**. The transitional section **422** may taper linearly or non-linearly along the length of the transitional section **422** (or any of the sections may taper). The second section **420** may further utilize a diffusion cooling hole to emit cooling fluid or gas from within the airfoil **400** at high velocity and cause the emitted cooling fluid or gas to wrap over the outer surface of the airfoil **400**. This creates a thin, protective film layer of cooling fluid or gas between the outer surface **405** of the airfoil **400** and the high temperature combustion gases. A diffusion cooling hole may be utilized with the airfoil passage **410** described herein, and the resulting outward cross-sectional area difference of the second section **420** does not detract from the heat transfer coefficient benefits of a decreasing taper of the first section **418** and the transitional section **422** of the airfoil passage **410**.

[0050] Cooling fluid or gas entering the first section **418** of the operating airfoil **400** may be relatively cool compared to the airfoil wall **401**. However, as cooling fluid or gas travels from first section **418** to the transitional section **422** and to the second section **420**, the cooling fluid or gas will gradually

increase in temperature. Therefore, in order to provide a constant amount of heat transfer throughout the length of the channel 416, the cooling fluid or gas flow in the second section 420 should travel at a higher velocity than the cooling fluid or gas flow through the first section 418. As a result, the cross-sectional area of second section 420 is smaller than the cross-sectional area of first section 418 to increase the velocity of cooling fluid or gas traveling through the airfoil passage 410.

[0051] Additionally, as shown in FIG. 4C, a first angle 424 is formed between the first section 418 and a corresponding inner surface 403 of the airfoil wall 401 (as shown in FIG. 4A), and may be between 15 and 90 degrees, and a second angle 426 is formed between the second section 420 and the outer surface 405 of the airfoil wall 400 (as shown in FIG. 4A), which may be between 15 and 90 degrees. The taper of the transitional section 422 may generally occur in the radial direction of the airfoil wall 401. However, the channel 416 may extend and/or taper in a radial and/or an axial direction of airfoil wall 401, or in another direction. Further, in FIG. 4C, the first section 418, the second section 420, and the transitional section 422 are shown generally in linear axial alignment. Alternatively, first section 418, second section 420, and transitional section 422 may be arranged in non-linearly.

[0052] The transitional section 422 may be oriented generally parallel to the airfoil wall 401 and may be further characterized by a ratio of transitional section length to airfoil wall width. The airfoil wall width may be defined as the thickness between the inner surface 403 of the airfoil wall 401 and the outer surface 405 of the airfoil wall 401. The transitional section length, fully enclosed within an airfoil wall in a generally axial direction, to airfoil wall width may be a minimum ratio of 3:1 to a maximum ratio dependent upon an airfoil span between the leading edge 402 and the trailing edge 404 of the airfoil 400.

[0053] FIG. 4D is a cross-sectional, perspective view of the airfoil passage 410 incorporated into the airfoil 400 shown in FIGS. 4A and 4B, in accordance with an embodiment of the present invention. In FIG. 4D, airfoil passage 410 includes a flow turbulator 428 within the airfoil passage 410. The flow turbulator 428 is shown as having a rectangular cross-section, but it is contemplated that the flow turbulator 428 may have any uniform or non-uniform shape optimized for increasing the rate of convective heat transfer between the airfoil 400 and the flow of cooling fluid or gas. Additionally, the flow turbulator 428 may comprise a plurality of flow turbulators 428 that may be arrayed in a linear or non-linear pattern within the airfoil passage 410, or may be integrally manufactured with the airfoil passage 410 to have a rough surface. In the disclosed embodiments, the heat transfer coefficient of the airfoil passage 410 may be additionally modified by tailoring the surface roughness of the interior of the airfoil passage 410 to achieve an equivalent roughness value of at least 400 Ra.

[0054] FIG. 5A is an angled, cross-sectional, perspective view of an airfoil 500 with variety of airfoil passages 510 integrated into an airfoil wall 501 of the airfoil 500, in accordance with an embodiment of the present invention. The airfoil 500 in FIG. 5A further comprises a leading edge airfoil passage 504 within the airfoil wall 501, which extends at least partially onto the sides of the airfoil 500.

[0055] The leading edge airfoil passage 504 includes at least one first opening 512 in the outer surface 505 of the airfoil wall 501, at least one second opening 514 in the outer surface 505 of the airfoil wall 501, and a channel 518 extend-

ing between the first opening 512 and the second opening 514 within the airfoil wall 501. The leading edge airfoil passage 504 further comprises at least one third opening 516 (which, in FIG. 5A, comprises two adjacent openings) in the inner surface 503 of the airfoil wall 501, which provides fluid communication between the channel 518 and an airfoil chamber 507 at least partially enclosed by the airfoil wall 501, through which cooling fluid or air may travel.

[0056] The cross-sectional area of the channel 518 is largest adjacent or proximate the third opening 516 at a third cross-sectional area 511 of the channel 518. The third opening 516, which may supply cooling fluid or gas from the airfoil chamber 507 to at least one of the first opening 512 and the second opening 514, and the third cross-sectional area 511 of the channel 518, is positioned proximate a stagnation region of high temperature corresponding to leading edge surface 502. This positioning of the third opening 516 within the channel 518, between first opening 512 and second opening 514 near the third cross-sectional area 511, allows the impingement effects of the third opening 516 to more effectively cool the airfoil wall 501.

[0057] The exemplary leading edge airfoil passage 504 may taper from the third cross-sectional area 511 axially and/or radially towards the first opening 512 and the second opening 514 within the leading edge 502 of the airfoil passage 504 in order to accelerate the flow of cooling fluid or gas passing through the leading edge airfoil passage 504. The leading edge airfoil passage 504 may be duplicated across the leading edge 502 of the airfoil 500 to provide enhanced cooling across the leading edge 502 of the airfoil 500 during operation of the gas turbine.

[0058] A first cross-sectional area of the first opening 512, which may be one of a plurality of first openings 512, referred to hereinafter as the first opening 512 for simplicity but intended to be non-limiting, of the leading edge airfoil passage 504 may be larger than a second cross-sectional area of the second opening 514, which may be one of a plurality of second openings 514, referred to hereinafter as the second opening 514 for simplicity but intended to be non-limiting, of the leading edge airfoil passage 504. The cross-sectional areas of the first opening 512 and second opening 514 are defined as the area between the walls of the channel at any position along the axial length of the channel. The leading edge airfoil passage 504 may be supplied with cooling fluid or gas from the airfoil chamber 507 through the third opening 516 in the inner surface 503 of the airfoil wall 501. The third opening 516, which may be one of a plurality of third openings 516, referred to hereinafter as the third opening 516 for simplicity but intended to be non-limiting, may further be referred to as an impingement hole. This cooling fluid or gas enters the airfoil wall 501 through the third opening 516, and then travels through the channel 518 towards the first opening 512 and the second opening 514 to exit the leading edge airfoil passage 504, carrying heat away from the airfoil wall 501.

[0059] The cross-sectional area of the channel 518 in the leading edge airfoil passage 504, as well as the other airfoil passages 510, may vary, linearly or non-linearly, across the length of channel 518, depending on the desired amount of heat transfer at different portions of the leading edge airfoil passage 504. In this respect, as shown in the leading edge airfoil passage 504, the cross-sectional area may be larger at the third cross-sectional area 511 of the channel 518 than at the first and second openings 512, 514, to allow acceleration

of cooling fluid or gas between the third opening **516** and the first and second openings **512**, **514** during cooling of the airfoil **500**.

[0060] FIG. 5B is a cross-sectional, elevation view of the airfoil **500** of FIG. 5A showing the plurality of airfoil passages **510** integrated therein, in accordance with an embodiment of the present invention. In FIG. 5B, as discussed with respect to FIG. 5A, the leading edge airfoil passage **504**, which may be repeated along the leading edge **502** of the airfoil **500**, may be supplied with cooling fluid or gas from the airfoil chamber **507** through the third opening **516**. This cooling fluid or gas travels through the leading edge **502** of the airfoil **500** by passing through the channel **518** to first opening **512** and second opening **514** to exit the airfoil wall **501**, carrying heat away from the airfoil **500**.

[0061] FIG. 6 depicts a cut-out, perspective view of the geometry of a plurality of leading edge airfoil passages **604** integrated into an airfoil **600**, in accordance with an embodiment of the present invention. FIG. 6 is used to representatively show the three-dimensional geometry of the leading edge airfoil passages **604** as they are arrayed on the leading edge **602** of the airfoil **600**. Furthermore, the leading edge airfoil passages **604** are connected via a plurality of connecting passages **609**. The connecting passages **609** provide fluid communication between each of the plurality of leading edge airfoil passages **604**. The connecting passages **609** may be positioned at any location along leading edge **602**, in order to provide the desired fluid communication between each of the plurality of leading edge airfoil passages **604**. Additionally, connecting passages **609** may be any shape, cross-sectional area, or frequency across the plurality of leading edge airfoil passages **604**.

[0062] FIGS. 7A-10C depict a variety of airfoil passage geometries **700**, **800**, **900**, **1000**, **1010**, and **1020** that can be integrated into an airfoil to provide enhanced cooling, in accordance with embodiments of the present invention. Referring now to FIGS. 7A-7C, a plurality of channels **702** having generally sharp-edged corners **704** are provided, in accordance with an embodiment of the present invention. The sharp-edged corners **704** are generally formed when two or more channels **702** having different angles intersect. Additionally, the intersections of channels **702** may be utilized to provide flow communication between the channels **702**. Cooling fluid or gas may be supplied through the channels **702** via impingement holes **706**. The cooling fluid or gas may then exit the channels **702** through openings **708** of the respective channels **702**. As previously discussed, the channels **702** may vary in cross-sectional area to control a velocity of cooling fluid or gas passing through the channels **702**.

[0063] FIG. 8 depicts a plurality of channels **802** and **803** in an alternate arrangement **800**, in accordance with an embodiment of the present invention. In FIG. 8, cooling fluid or gas may be supplied to the channels **802** and **803**, with the channels separated by a dividing portion **801**. More specifically, the cooling fluid or air may be supplied to the channels **802** and **803** through a plurality of impingement holes **808**, such that the cooling fluid or gas passes through the channels **802** and **803** towards respective first and second openings **810** and **811**. In FIG. 8, a plurality of turbulators **804** are shown along the length of a side-wall **806** of the channels **802**, **803**. The plurality of turbulators **804** are shown in FIG. 8 as having a rectangular cross-sectional shape. However, it is contemplated that the plurality of turbulators **804** may have other cross-sectional shapes, including asymmetrical or non-

form shapes, or integrally manufactured leading edges having a rough surface. In the disclosed embodiments, the leading edge channel heat transfer coefficient may be modified by additionally tailoring the surface roughness to achieve an equivalent roughness of at least 400 Ra.

[0064] As shown in FIG. 8, the plurality of turbulators **804** are arrayed in a parallel pattern along a length of the channel **802**. However, the plurality of turbulators **804** may be patterned in a non-parallel pattern as well, in order to alter the fluid dynamics in the channels **802**. For instance, the turbulators **804** may comprise multiple rows of turbulators. Additionally, each row of turbulators **804** may be angled with respect to the channel **802** (and any other channels **802**, **803** into which it is integrated). Further, turbulators **804** may be positioned at any location within channels **802** and **803**, and are not limited to a row configuration.

[0065] Referring now to FIGS. 9A and 9B, a plurality of tapered channels **902** in an alternate arrangement **900** which may be integrated into a leading edge of an airfoil is provided, in accordance with an embodiment of the present invention. In operation, cooling fluid or gas may be provided to the channels **902** through impingement holes **904** shown in FIGS. 9A and 9B. As cooling fluid or gas passes into the channels **902** from the impingement holes **904**, the cooling fluid or gas accelerates towards respective first openings **905** and respective second openings **907** of the channels **902** along a side wall **906** due to the narrowing of the channels **902** towards the openings **905**, **907**.

[0066] Referring now to FIGS. 10A-10C, alternate arrangements **1000**, **1010**, and **1020** of exemplary airfoil passages are depicted, in accordance with embodiments of the present invention. The arrangements **1000**, **1010**, and **1020** generally comprise different embodiments of a wave-like channel **1002**, which may be incorporated into a leading edge region of an airfoil. The wave-like channel **1002**, as shown in FIGS. 10A-10C, may comprise a first portion **1003** at a first angle, a second portion **1005** at a second angle, and a rounded transitional portion **1007** which connects the first and the second portions **1003**, **1005**. This rounded transitional portion **1007** creates the rounded "hill and valley" design effect shown in FIGS. 10A-10C. Such a pattern may be repeated throughout the wave-like channels **1002**. In operation, cooling fluid or gas may be provided to the plurality of channels **1002** through impingement holes **1004**. As with prior designs, the channels **1002** may decrease in cross-sectional area from the respective impingement holes **1004** to respective first and second openings **1008**, **1009**.

[0067] Referring now to FIG. 11, a block diagram of an exemplary method **1100** of manufacturing airfoils is provided, in accordance with an embodiment of the present invention. At block **1110**, an airfoil, such as the airfoil **500** depicted in FIG. 5A, is provided. The airfoil comprises an airfoil wall, such as the airfoil wall **501** shown in FIG. 5A, including an inner surface, such as the inner surface **503** shown in FIG. 5A, and an outer surface, such as the outer surface **505** shown in FIG. 5A, the airfoil wall forming an airfoil chamber, such as the airfoil chamber **507** shown in FIG. 5A, at least partially enclosed within the airfoil wall.

[0068] At block **1120**, a plurality of airfoil passages, such as the leading edge airfoil passage **504** shown in FIG. 5A, are formed at a leading edge, such as the leading edge **502** of the airfoil **500** shown in FIG. 5A, of the airfoil wall. As discussed herein, each of the plurality of airfoil passages comprises a first opening, such as the first opening **512** shown in FIG. 5A,

in the outer surface, a second opening, such as the second opening **514** shown in FIG. **5A**, in the outer surface, and a channel, such as the channel **518** shown in FIG. **5A**, extending from at least one of the first opening and the second opening to a third opening, such as the third opening **516** shown in FIG. **5A**, the third opening providing fluid communication between the channel and the airfoil chamber.

[0069] The plurality of airfoil passages may be formed using additive manufacturing, such as selective laser melting (SLM), or another method. The first opening may include a first cross-sectional area and the second opening may include a second cross-sectional area, the first cross-sectional area being larger than the second cross sectional area.

[0070] Referring now to FIG. **12**, a block diagram of another exemplary method **1200** of manufacturing airfoils is provided, in accordance with an embodiment of the present invention. At block **1210**, an airfoil, such as the airfoil **500** depicted in FIG. **5A**, is provided. The airfoil comprises an inner surface, such as the inner surface **503** shown in FIG. **5A**, and an outer surface, such as the outer surface **505** shown in FIG. **5A**, such that the airfoil wall forms an airfoil chamber, such as the airfoil chamber **507** shown in FIG. **5A**, at least partially enclosed within the airfoil wall. At block **1220**, a plurality of airfoil passages, such as the airfoil passages **510** shown in FIG. **5A**, are formed within the airfoil wall. Each of the airfoil passages comprises at least one first opening, such as the first opening **512** shown in FIG. **5A**, in the inner surface, at least one second opening, such as the second opening **514** shown in FIG. **5A**, in the outer surface, and a channel, such as the channel **518** shown in FIG. **5A**, extending from the first opening to the second opening. The channel decreases in cross-sectional area between the at least one first opening and the at least one second opening. The plurality of airfoil passages may be formed at least partially in a leading edge wall of the airfoil, and/or at least partially on a pressure side wall and a suction side wall of the airfoil.

[0071] Referring now to FIG. **13**, a block diagram of another exemplary method **1300** of manufacturing airfoils is provided, in accordance with an embodiment of the present invention. At block **1310**, an airfoil, such as the airfoil **500** shown in FIG. **5A**, having a leading edge, such as the leading edge **502** shown in FIG. **5A**, and a trailing edge, such as the trailing edge **404** shown in FIG. **4A**, is provided. The airfoil comprises an airfoil wall, such as the airfoil wall **501** shown in FIG. **5A**, having an inner surface, such as the inner surface **503** shown in FIG. **5A**, and an outer surface, such as the outer surface **505** shown in FIG. **5A**, the airfoil wall forming an airfoil chamber, such as the airfoil chamber **507** shown in FIG. **5A**, at least partially enclosed by the airfoil wall.

[0072] At block **1320**, a plurality of pockets, such as the pockets **310**, **312**, **314**, and **316** shown in FIG. **3A**, are formed within the airfoil wall. Each of the plurality of pockets comprises an inner pocket wall, such as the inner pocket wall **324** shown in FIG. **3A**, and an outer pocket wall, such as the outer pocket wall **326** shown in FIG. **3A**. Additionally, a first opening, such as the first opening **318** shown in FIG. **3A**, may be positioned in the inner surface at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket, and a second opening, such as the second opening **320** shown in FIG. **3A**, may be positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the pocket. Further, a distance between the inner pocket wall and the outer pocket

wall is larger proximate the leading edge of the airfoil and smaller approximate the trailing edge of the airfoil.

[0073] From the foregoing, it will be seen that this invention is one well adapted to attain all the ends and objects hereinabove set forth together with other advantages which are obvious and which are inherent to the structure. It will be understood that certain features and subcombinations are of utility and may be employed without reference to other features and subcombinations. This is contemplated by and is within the scope of the claims. Since many possible embodiments may be made of the invention without departing from the scope thereof, it is to be understood that all matter herein set forth or shown in the accompanying drawings is to be interpreted as illustrative and not in a limiting sense. Additional objects, advantages, and novel features of the invention will be set forth in part in the description which follows, and in part will become apparent to those skilled in the art upon examination of the following, or may be learned by practice of the invention.

What is claimed is:

1. An airfoil for a gas turbine having a leading edge and a trailing edge, the airfoil comprising:
 - an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall; and
 - a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising:
 - an inner pocket wall and an outer pocket wall;
 - at least one first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket; and
 - at least one second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the pocket,
- wherein a distance between the inner pocket wall and the outer pocket wall is larger proximate the leading edge and smaller proximate the trailing edge.
2. The airfoil of claim 1, wherein each of the plurality of pockets further comprises a plurality of pedestals, wherein the plurality of pedestals extend between the inner pocket wall and outer pocket wall of each respective pocket of the plurality of pockets.
3. The airfoil of claim 2, wherein each pedestal of the plurality of pedestals of each of the plurality of pockets has a circular, triangular, square, ovular, or rectangular cross-section.
4. The airfoil of claim 2, wherein a distance between each of the plurality of pedestals in each of the plurality of pockets is larger proximate the leading edge and smaller proximate the trailing edge.
5. The airfoil of claim 2, wherein the plurality of pockets are arrayed in a linear or non-linear pattern within the airfoil wall.
6. The airfoil of claim 2, wherein the plurality of pockets within the airfoil wall and the plurality of pedestals are formed using additive manufacturing.
7. The airfoil of claim 1, wherein a ratio of cross-sectional area between the inner pocket wall and the outer pocket wall of each of the plurality of pockets is between 1.1:1 and 10:1.

8. The airfoil of claim **7**, wherein for each of the plurality of pockets, the inner pocket wall and the outer pocket wall are substantially parallel to the inner surface of the airfoil wall.

9. The airfoil of claim **1**, wherein for each of the plurality of pockets, the second opening is positioned on a pressure side wall or a suction side wall of the airfoil.

10. The airfoil of claim **1**, wherein a ratio of pocket axial length to airfoil wall width in each of the plurality of pockets is at least 3:1.

11. The airfoil of claim **1**, wherein the plurality of pockets are arrayed in a radial direction within the airfoil wall, and wherein a ratio of pocket radial height to airfoil wall width in each of the plurality of pockets is at least 3:1.

12. A gas turbine assembly, the assembly comprising:
an airfoil having a leading edge and a trailing edge, the airfoil comprising an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall; and

a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising:

an inner pocket wall and an outer pocket wall;
at least one first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket; and

at least one second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the respective pocket,

wherein a distance between the inner pocket wall and the outer pocket wall is larger proximate the leading edge and smaller proximate the trailing edge, and

wherein a ratio of cross-sectional area of the inner pocket wall to the outer pocket wall is between 1.1:1 and 10:1.

13. The assembly of claim **12**, wherein each of the plurality of pockets further comprises a plurality of pedestals that extend between the inner pocket wall and the outer pocket wall.

14. The assembly of claim **13**, wherein each pedestal of the plurality of pedestals of each of the plurality of pockets has a circular, triangular, square, ovular, or rectangular cross-section.

15. The assembly of claim **14**, wherein a distance between each of the plurality of pedestals in each of the plurality of pockets is larger proximate the leading edge and smaller proximate the trailing edge.

16. The assembly of claim **12**, wherein the plurality of pockets within the airfoil wall are arrayed in a linear or a non-linear pattern.

17. The assembly of claim **12**, wherein the inner pocket wall and the outer pocket wall of each of the plurality of pockets are substantially parallel to the inner surface of the airfoil wall.

18. The assembly of claim **12**, wherein the second opening is positioned in a pressure side wall, a suction side wall, or the trailing edge of the airfoil.

19. The assembly of claim **12**, wherein a ratio of pocket axial length to airfoil wall width in each of the plurality of pockets is at least 3:1.

20. The assembly of claim **12**, wherein a ratio of pocket radial height to airfoil wall width in each of the plurality of pockets is at least 3:1.

21. A method of manufacturing airfoils, the method comprising:

providing an airfoil having a leading edge and a trailing edge, the airfoil having an airfoil wall having an inner surface and an outer surface, the airfoil wall forming an airfoil chamber at least partially enclosed by the airfoil wall; and

forming a plurality of pockets within the airfoil wall, each of the plurality of pockets comprising:

an inner pocket wall and an outer pocket wall;
a first opening in the inner surface positioned at a first distance away from the leading edge, the first opening providing fluid communication between the airfoil chamber and the pocket; and

a second opening in the outer surface positioned at a second distance away from the leading edge, the second opening providing fluid communication between an outside of the airfoil and the pocket,

wherein a distance between the inner pocket wall and the outer pocket wall is larger proximate a leading edge of the airfoil and smaller proximate a trailing edge of the airfoil.

22. The method of claim **21**, wherein the pocket is formed using additive manufacturing.

23. The method of claim **21**, wherein a ratio of pocket axial length to airfoil wall width for each of the plurality of pockets is at least 3:1.

24. The method of claim **21**, wherein a ratio of pocket radial height to airfoil wall width for each of the plurality of pockets is at least 3:1.

25. The method of claim **21**, wherein each of the plurality of pockets further comprises a plurality of pedestals, wherein the plurality of pedestals extend between the inner pocket wall and outer pocket wall of each respective pocket of the plurality of pockets.

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