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(54) **HEAT STORAGE DEVICE FOR AN ENGINE**

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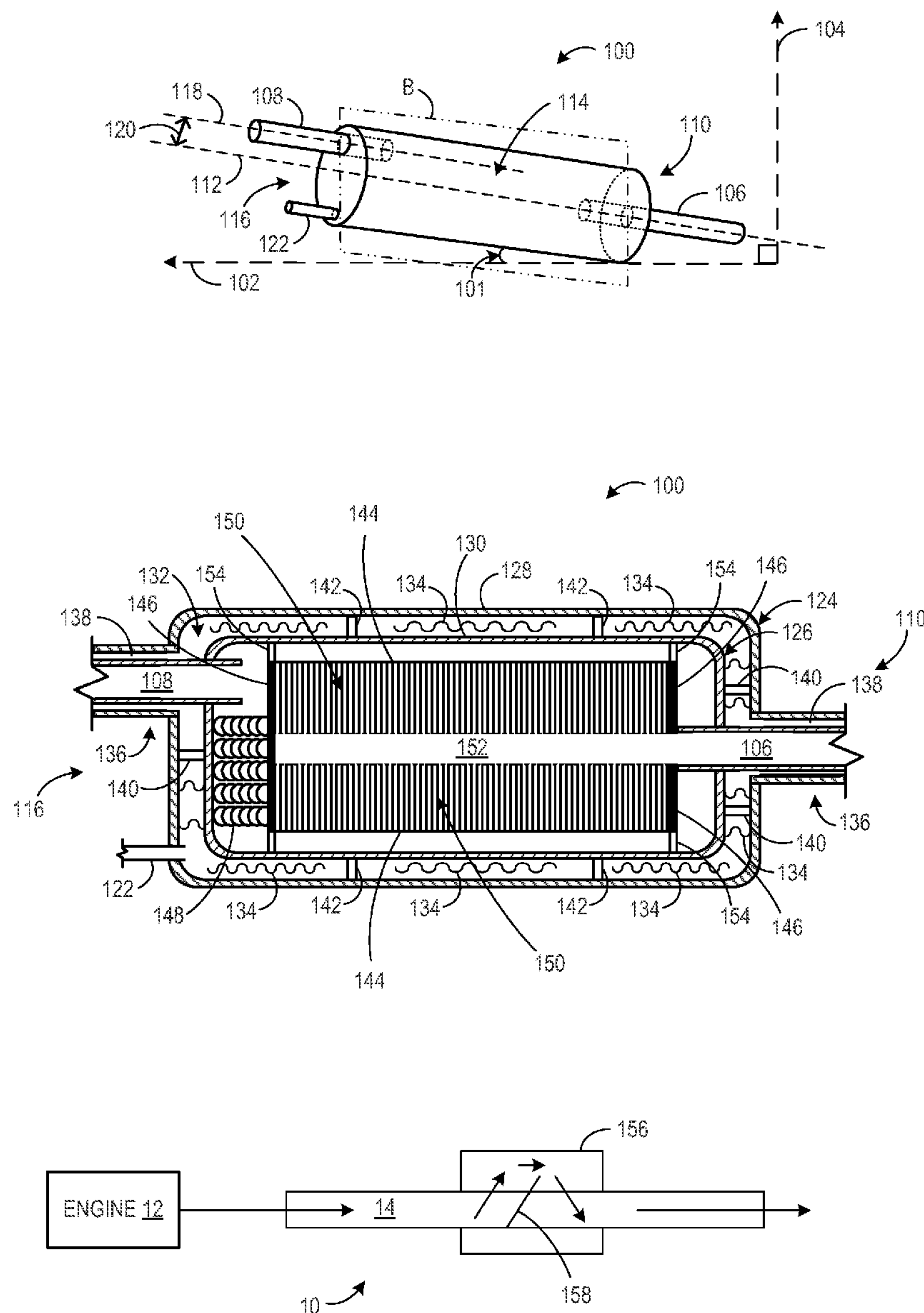
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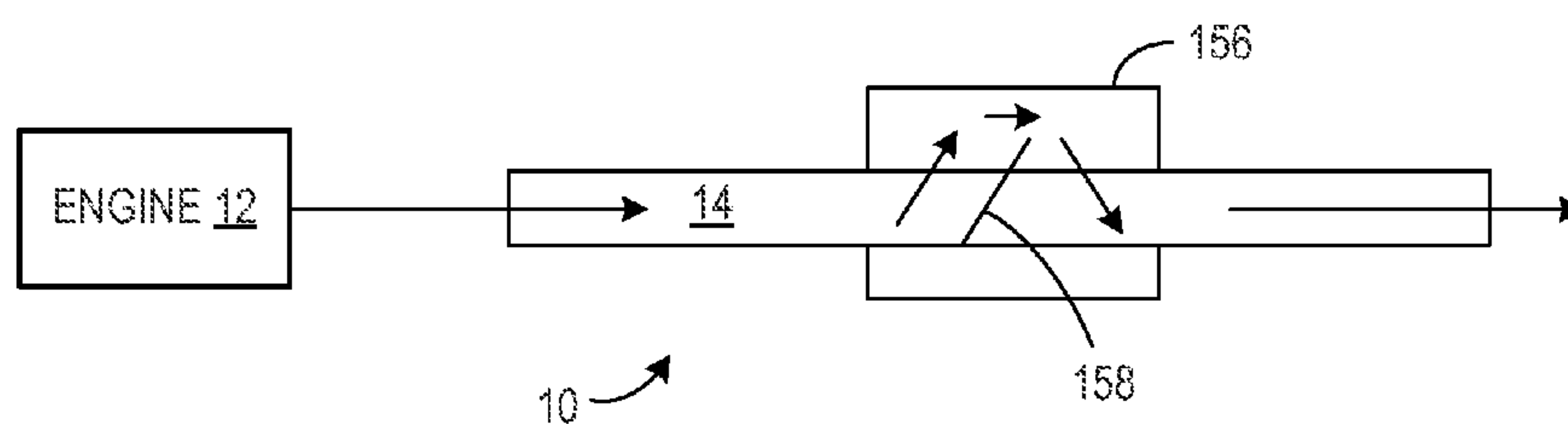
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(57) **ABSTRACT**

A heat recovery system for an engine is disclosed herein utilizing a heat storage unit configured to store waste exhaust heat for subsequent use on engine starts in various systems. In this way, improved system operation can be obtained by re-using such waste heat.





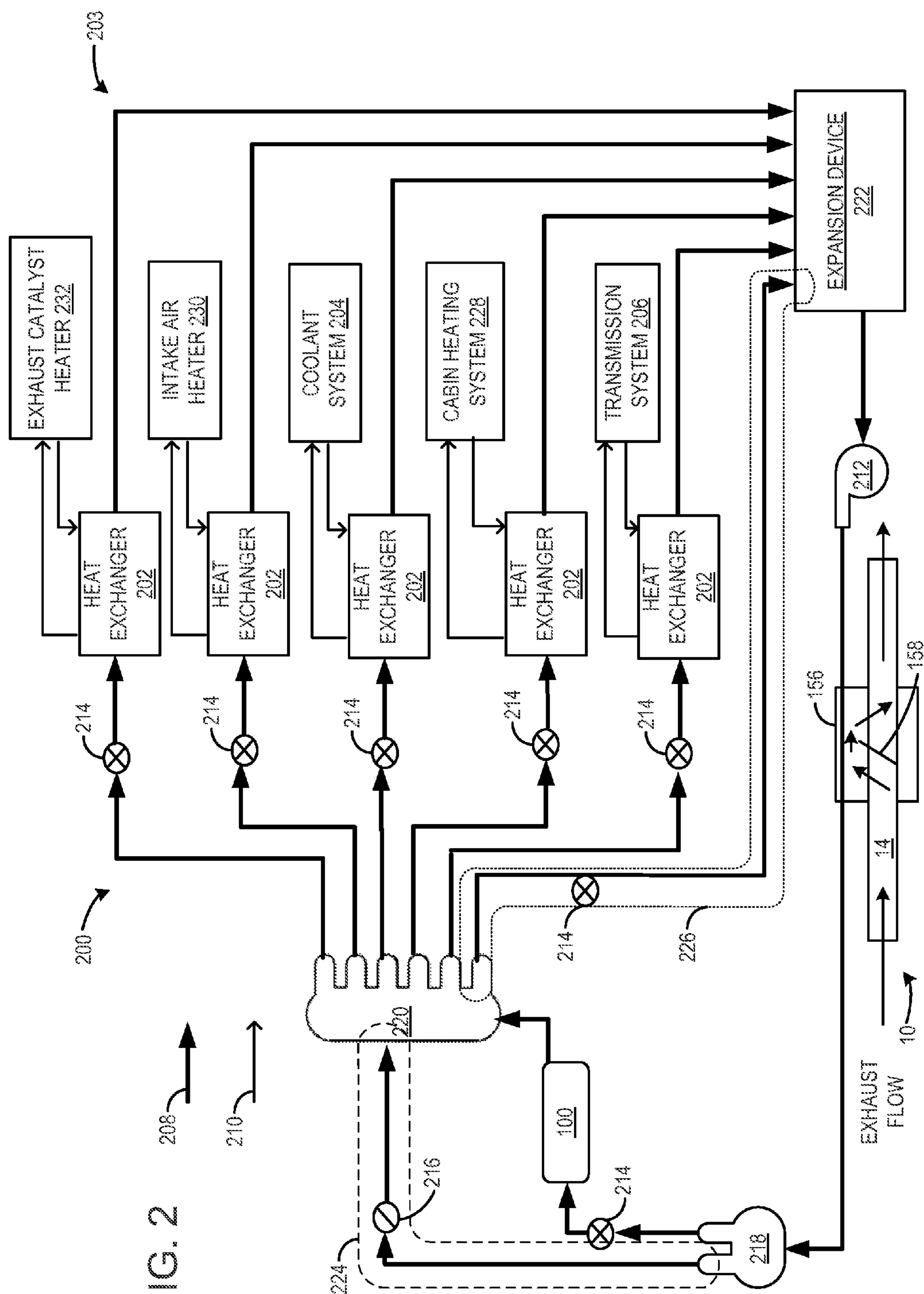
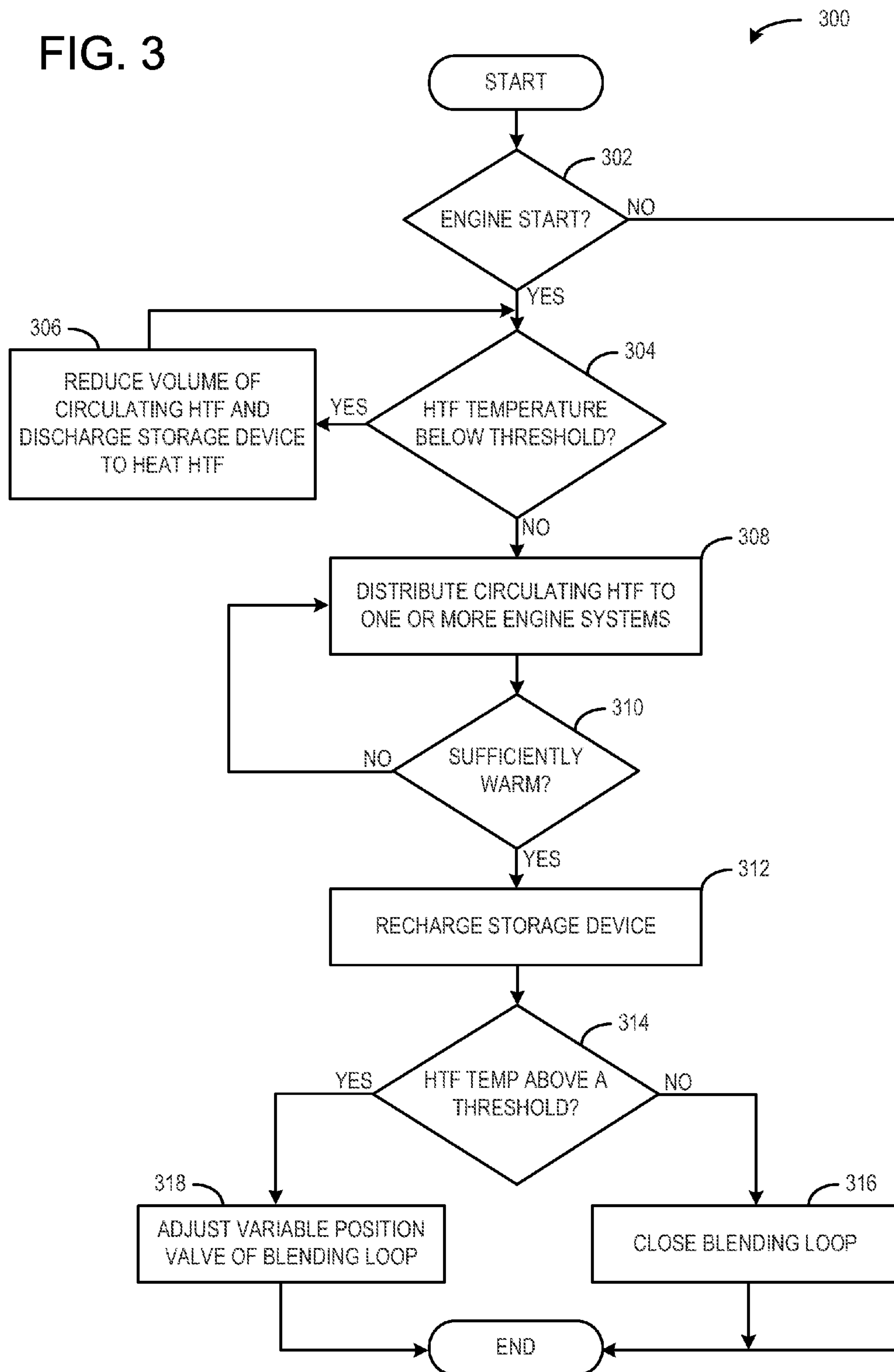
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FIG. 3



HEAT STORAGE DEVICE FOR AN ENGINE

BACKGROUND AND SUMMARY

[0001] Vehicles may recover exhaust heat for transfer to various other systems in an internal combustion engine.

[0002] The inventors herein have recognized various issues with such systems in that the available packaging space for storing waste heat is limited, and further suitable coordination among various vehicle components is lacking. As such, one example approach to address the above issues is, claim 1.

[0003] It should be understood that the summary above is provided to introduce in simplified form a selection of concepts that are further described in the detailed description. It is not meant to identify key or essential features of the claimed subject matter, the scope of which is defined uniquely by the claims that follow the detailed description. Furthermore, the claimed subject matter is not limited to implementations that solve any disadvantages noted above or in any part of this disclosure.

BRIEF DESCRIPTION OF THE DRAWINGS

[0004] FIGS. 1A-1B schematically show an example heat storage device that may be included in an exhaust system.

[0005] FIG. 1C schematically shows an example heat exchanger that may be coupled to the heat storage device of FIGS. 1A-1B.

[0006] FIG. 2 schematically shows an example heat recovery system including the heat storage device of FIGS. 1A-1B.

[0007] FIG. 3 schematically shows an example method for operating the heat recovery system of FIG. 2.

DETAILED DESCRIPTION

[0008] The following description relates to a heat storage device of a heat transfer system including phase changing materials, which are arranged in such a way that thermal energy from an exhaust system can be recovered. The example arrangements described herein allow thermal energy to be recovered and stored for later heating of a passenger compartment, for example.

[0009] As indicated, the heat transfer system may utilize a heat storage device to transfer heat even when the engine is not in operation. For example, the heat storage device may be in fluidic communication with an exhaust system component downstream from the catalytic converter, such as via heat exchanger. In this way, heat may transfer from the heat storage device even after the engine is no longer in operation. For example, the heat storage device may be insulated to store heat recovered from the exhaust system, which may be available for immediate use at engine start.

[0010] Additionally, the heat transfer system may include various heat transfer fluids to extract thermal energy from the exhaust system under a variety of different operating conditions. In this way, thermal energy may be recovered from the exhaust system to provide heat to various other systems such as a cabin heating system, lubrication systems, and/or other exhaust system components, if desired.

[0011] Further, the example systems allow for a simpler and more compact design as compared to traditional designs. For example, the heat storage device may provide heat to a cabin heating system at engine start, as introduced above. By coupling the heat storage device to a component of the exhaust system downstream from the catalytic converter, the cabin heating system may provide heat to the passenger cabin

at engine start without relying upon a coolant system, and therefore, without waiting for the coolant system to warm up at engine start. Further, the system may provide the stored heat to the cabin heating system without delaying catalytic converter light-off, as described above.

[0012] FIGS. 1A and 1B show a heat storage device 100 according to an embodiment of the present disclosure. FIG. 1A shows a perspective exterior view and FIG. 1B shows a perspective cross sectional view of heat storage device 100 taken along plane B. FIG. 1 shows an example bypass heat exchanger that may be coupled to heat storage device 100.

[0013] Referring first to FIG. 1A, heat storage device 100 may be a cylindrical shape. In other words, heat storage device 100 may have a circular cross section. Further, heat storage device 100 may be tilted by an angle 101 from a horizontal. Such an angle may facilitate efficient heat transfer through the device. As used herein, the horizontal refers to the ground over which the vehicle travels. For example, FIG. 1A shows a horizontal axis 102 and a vertical axis 104. The vertical axis may be orthogonal to the horizontal axis. Therefore, the vertical axis may be orthogonal to the ground over which the vehicle travels. As shown, heat storage device 100 may be tilted angle 101 from horizontal axis 102. In some embodiments, heat storage device 100 may be tilted 5° from the horizontal; however, it will be appreciated that other angles are possible without departing from the scope of this disclosure. Further, in some embodiments, heat storage device 100 may not be tilted. For example, heat storage device 100 may be level with the horizontal. In other words, angle 101 may be zero degrees.

[0014] As shown, heat storage device 100 includes an inlet passage 106 and an outlet passage 108. The inlet and outlet passages may carry a heat transfer fluid. Further, the heat storage device may house a phase changing material (PCM).

[0015] Inlet passage 106 may be coupled to heat storage device 100 at a central position. For example, inlet passage 106 may be coupled to a first end 110 of heat storage device 100 at the central position. In other words, inlet passage 106 may have a central axis 112 that is shared with a central axis of end 110, and further, shared with a central axis of heat storage device 100. Inlet passage 106 may be configured to supply heat storage device 100 with heat recovered from the exhaust system, for example. In some embodiments, the heat transfer fluid of inlet passage 106 may be coupled to a pump (not shown) to drive a movement of the heat transfer fluid. Further, a bypassable heat exchanger may be positioned upstream from inlet passage 106. Such a heat exchanger is discussed further with reference to FIG. 1C. Further, inlet passage 106 may include a portion that extends into an interior 114 of heat storage device 100.

[0016] Outlet passage 108 may be coupled to heat storage device 100 at a top position. For example, outlet passage 108 may be coupled to a second end 116 of heat storage device 100 at the top position. In other words, outlet passage 108 may have a central axis 118 that is a distance 120 from shared central axis 112 in a vertical direction (e.g., along vertical axis 104). In this way, outlet passage 108 is positioned towards a periphery of end 116, rather than centrally located, to advantageously reduce bubble accumulation in the heat transfer fluid. However, in some embodiments, outlet passage 108 may be centrally located at end 116, if desired. Outlet passage 108 may be configured to transfer heat from heat storage device 100 to another system of the vehicle. For example, outlet passage 108 may transfer stored heat to the cabin heat-

ing system, the coolant system, the lubrication system, and/or another system of the vehicle. Further, outlet passage 108 may include a portion that extends into the interior 114 of heat storage device 100.

[0017] As shown, heat storage device 100 includes a vacuum passage 122. For example, vacuum passage 122 may be coupled to heat storage device 100 at end 116. Vacuum passage 122 may be coupled to both heat storage device 100 and a vacuum pump (not shown). For example, in some embodiments, heat storage device 100 may include a vacuum jacket, and vacuum passage 122 may be a conduit for evacuating an airspace within the vacuum jacket. In this way, a pressure within at least a portion of interior 114 may be reduced. In some embodiments, the pressure within interior 114 may be reduced to 1 microbar or less.

[0018] FIG. 1B shows a perspective interior view of heat storage device 100. As shown, heat storage device 100 may be double walled. In other words, heat storage device 100 may include an outer vessel 124 and an inner vessel 126. For example, heat storage device may include outer walls 128 and inner walls 130. Further, heat storage device may include vacuum jacket 132 positioned between outer walls 128 and inner walls 130. As described above, vacuum passage 122, along with a vacuum pump, may suction air out of vacuum jacket 132 such that a pressure within vacuum jacket 132 is reduced.

[0019] Vacuum jacket 132 may hold a reduced pressure around an exterior of inner vessel 126 when a vacuum is applied. By applying a vacuum, water vapor and other gaseous compounds can be evacuated from the surfaces of the insulating layers as hot fluid is pumped through the heat transfer fluid passages. Further, vacuum jacket 132 may include one or more anti-radiation foils 134 that reduce heat loss to the surrounding environment via radiation.

[0020] It will be appreciated that the perspective view of FIG. 1B shows a longitudinal cross section of the heat storage device 100, thus it will be appreciated that outer walls 128, inner walls 130, and vacuum jacket 132 extend circumferentially around a perimeter of heat storage device 100 and longitudinally, for example, along axis 112.

[0021] Further, at least a portion of inlet passage 106 and outlet passage 108 may be double walled and include a vacuum space. For example, portions 136 exterior to heat storage device 100 may be double walled similar to the inner and outer vessels. Further, vacuum spaces 138 of the inlet and outlet passages may coalesce with vacuum jacket 132 of the heat storage device.

[0022] Heat storage device 100 may include one or more axial supports 140. Axial supports 140 may couple inner vessel 126 to outer vessel 124 such that the inner vessels is suspended and supported within the outer vessel. As shown, axial supports 140 may be coupled to outer walls 128 and inner walls 130, and thus, may be positioned within vacuum jacket 132. The axial supports may be composed of a material with low heat conducting properties. For example, axial supports 140 may be composed of titanium or a composite including titanium or another material with low heat conducting properties. Further, in some embodiments the axial supports may be perforated to further reduce heat loss to the surrounding environment.

[0023] Further, the inner vessel may be additionally and/or alternatively supported by radial supports 142. Such radial supports may be located circumferentially at various positions. As shown, radial supports 142 may be coupled to outer

walls 128 and inner walls 130, and thus, may be positioned within vacuum jacket 132. Similar to the axial supports, the radial supports 142 may be composed of titanium or a composite including titanium or another material with low heat conducting properties. Further, in some embodiments the radial supports may be perforated to further reduce heat loss to the surrounding environment.

[0024] As shown, heat storage device 100 includes two axial supports at end 110, one axial support at end 116, and four radial supports 142. It will be appreciated that the number of axial and radial supports shown is non-limiting and another number of supports and/or another configuration of supports is possible without departing from the scope of this disclosure. The supports are provided to illustrate a general concept of a configuration enabling heat storage device 100 to withstand gravitational acceleration forces that may occur when the heat storage device 100 is rigidly coupled to the vehicle body.

[0025] Heat storage device 100 may further include a phase changing material (PCM) stack 144 supported between retention plates 146 via one or more springs 148. PCM stack 144 may include a plurality of PCM elements 150 arranged radially about a central feed passage 152. In some embodiments, the configuration of the PCM stack is such that the PCM stack retains 80% of stored heat for at least 16 hours, which may be used as a heat source at engine start to heat the passenger cabin, as described above. Further, heat stored in PCM stack 144 may be discharged to heat the passenger cabin or another engine system without starting the engine. For example, PCM stack discharge may be initiated remotely and does not necessarily have to coincide with engine-start. However, PCM stack discharge may be initiated remotely along with engine-start, for example, using a remote starter to start engine 12.

[0026] The plurality of PCM elements 150 include a phase changing material capable of storing a large quantity of heat in a form of a latent heat of fusion. Since the plurality of PCM elements 150 are surrounded by the double wall configuration, heat storing capabilities are enhanced. In other words, the double wall configuration acts like a thermos to retain heat stored within the plurality of PCM elements 150. In some embodiments, each PCM element may include the same phase changing material, and thus, the PCM stack may have one phase transfer temperature. In other embodiments, the PCM stack may include PCM elements with different phase changing materials, wherein each different phase changing material has a different phase transfer temperature. In such an example, a time to charge the PCM stack may be reduced. In other words, the time for the PCM stack to reach a maximum heat storing potential may be reduced.

[0027] As shown, heat transfer fluid may be delivered to PCM stack 144 via centrally located inlet passage 106, and further, via center feed passage 152. Thus, it will be appreciated that inlet passage 106 is in fluidic communication with center feed passage 152. Thus, heat transfer fluid flows radially from center feed passage 152 to the plurality of PCM elements 150. Heat transfer fluid exits the heat storage device via outlet passage 108 arranged in the top position, as described above.

[0028] As the heat transfer fluid flows through the PCM stack, a pressure drop occurs. To reduce the pressure drop, the inlet passage 106 and the outlet passage 108 are straight. In other words, the inlet passage 106 and the outlet passage 108 do not include bends. Further, the inlet passage 106 and the outlet passage 108 do not include corrugations. Due to the

absence of corrugations, a rate of heat loss may potentially increase. However, since the inlet and outlet passages include a vacuum space around a circumference of these passages, such a potential for heat loss is reduced.

[0029] As shown, retention plates **146** may be positioned at either end of PCM stack **144**. For example, one retention plate **146** may be positioned proximate to end **110**, and another retention plate **146** may be positioned proximate to end **116**. Retention plates **146** may be a circular shape and may have a diameter that is approximately equal to a diameter of PCM stack **144**. As another example, retention plates **146** may have a larger diameter or a smaller diameter than PCM stack **144**. The retention plates may be coupled to the inner vessel via one or more plate extensions with windows **154** to allow HTF to reach exit **108**. Six axial rods (not shown) allow retention of the PCM stack in the radial and circumferential directions. The rods are welded to the retention plates. As such, the PCM stack is retained inside inner vessel **126** to reduce the potential for stack element sliding and/or rotation during vehicle operation.

[0030] Further, one or more springs **148** may further maintain the position of the PCM stack. As shown, one or more springs **148** may be positioned proximate to end **116** between retention plate **146** and inner walls **130**. Springs **148** may be configured to ensure proper contact between the PCM elements during thermal expansion and thermal compression that results from the heat transfer fluid heating and cooling. In some embodiments, springs **148** may have a combined force of 100 Newtons or higher to maintain proper contact between the PCM elements. As shown in FIG. 1B, heat storage device **100** may include five springs; however the heat storage device may include more than five springs or less than five springs, if desired.

[0031] FIG. 1C schematically shows a heat exchanger **156** thermally coupled to exhaust passage **14**. In some embodiments, heat exchanger **156** may be thermally coupled to exhaust passage **14** at a position between engine **12** and one or more exhaust emission control devices **16**. For example, heat exchanger **156** may be thermally coupled to exhaust passage **14** upstream from an oxidation catalyst such as a diesel oxidation catalyst (DOC). Heat exchanger **156** may be fluidically coupled to heat storage device **100** via inlet passage **106**. For example, heat exchanger **156** may be thermally coupled to inlet passage **106** at a position upstream from heat storage device **100**. In some embodiments, heat exchanger **156** may be an evaporative region to extract heat from heat passage **14** and provide said heat to heat storage device **100** by way of inlet passage **106**. For example, heat exchanger **156** may include heat transfer tubing that carries heat transfer fluid supplied by an engine-driven pump to flow inside the tubing. As another example, the heat transfer tubing may carry heat transfer fluid supplied by an electrically-driven pump to flow inside the tubing. Such a configuration of heat transfer tubing may be fluidically coupled to inlet passage **106**. As one example, the heat transfer fluid of the tubing may be the same heat transfer fluid of inlet passage **106**. As another example, the heat transfer fluid of the tubing may be a different fluid than the heat transfer fluid of inlet passage **106**.

[0032] In some embodiments, heat exchanger **156** is a liquid-to-liquid heat exchanger. In other embodiments, the heat exchanger **156** could be a gas-to-liquid or gas-to-thermosyphon heat exchanger.

[0033] Further, exhaust passage **14** may include a bypass valve **158** that directs exhaust gas flow through heat

exchanger **156**. Bypass valve **158** is shown in a bypass position (e.g., a closed position) in FIG. 1C. Bypass valve **158** may be actuated via a controller, or bypass valve **158** may be a passive valve, if desired. Bypass valve **158** may be in an open position (e.g., exhaust gases are not diverted to heat exchanger **156**) when exhaust back pressure reaches a threshold value. For example, bypass valve **158** may be closed at high exhaust flows and/or high exhaust temperatures. As such, bypass valve **158** may reduce loss of engine output, and therefore, may reduce fuel consumption.

[0034] It will be appreciated that the disclosed system may include more than one heat exchanger. For example, a heat exchanger may be positioned upstream from inlet passage **106**, and one or more heat exchangers may be positioned downstream from outlet passage **108**. For example, a heat exchanger may be positioned at an interface between the heat recovery system and another system of the vehicle. Such a configuration is described in further detail with respect to FIG. 2.

[0035] It will be appreciated that FIGS. 1A-1C are shown in simplified form and that numerous variations are possible without departing from the scope of this disclosure. Further, heat storage device **100** may include additional and/or alternative components than those illustrated in FIGS. 1A and 1B. Further still, it is to be understood that heat storage device **100** is provided to illustrate a general concept, and thus, numerous geometric configurations are possible without departing from the scope of this disclosure.

[0036] FIG. 2 schematically shows a heat recovery system **200** including heat storage device **100** and a plurality of heat exchangers. FIG. 2 includes similar features as FIG. 1, and like features are indicated with common reference numbers. Such features will not be discussed repetitively for the sake of brevity.

[0037] As shown, heat recovery system **200** includes heat exchanger **156** to recover heat from exhaust system **10**, as described above. Heat recovery system **200** may further include one or more additional heat exchangers **202**. Heat exchangers **202** may transfer heat between heat recovery system **200** and another engine system **203**. For example, heat exchangers **202** may transfer heat to coolant system **204**, cabin heating system **228**, and/or transmission system **206**. In other words, heat exchangers **202** may be thermally coupled (e.g., in thermal contact) with a fluid of the coolant system **204**, the cabin heating system **228** and/or the transmission system **206** to transfer heat to each respective system.

[0038] It will be appreciated that each of the engine systems **203** are separate systems from heat recovery system **200** and exhaust system **10**. As such, engine systems **203** include components that are separate from the components of heat recovery system **200** and exhaust system **10**. Thus, engine systems **203** do not include heat exchanger **156**, heat exchangers **202**, heat storage device **100**, or another component of heat recovery system **200** and exhaust system **10**. For example, cabin heating system **228** may include a heater core and a fan, wherein the heater core and the fan are separate from the heat recovery system and the exhaust system. Thus it is to be understood that only a fluid conduit (e.g., a coolant passage) of each engine system **203** is in thermal contact with the heat recovery system **200** at a position coinciding with the heat exchanger **202**, for example. In this way, heat transfer occurs at the heat exchanger.

[0039] It will be appreciated that one or more of the heat exchangers may be gas-to-liquid and/or gas-to-thermosy-

phon heat exchangers. As shown, heat exchangers **202** may be thermally coupled to an engine system in parallel. In some embodiments, heat exchangers **202** may be thermally coupled to each of the engine systems in series. For example, heat transfer fluid may flow through a series of heat exchangers **202** fluidically coupled to a common heat transfer fluid passage.

[0040] As shown, heat transfer fluid (HTF) may flow through a heat exchanger and may thermally transfer heat to a fluid of one or more of the aforementioned systems. Arrows **208** generally indicate a direction of HTF flow, and arrows **210** generally indicate a direction of fluid flow for each engine system. Pump **212** may drive HTF fluid flow through heat recovery system **200**. As shown, pump **212** is positioned upstream from heat exchanger **156**; however, another position is possible without departing from the scope of this disclosure. Further, it will be appreciated that coolant system **204**, cabin heating system **228** and/or the transmission system **206** may have another driving mechanism to drive fluid flow through each respective system. For example, each engine system **203** may have a pump, similar to pump **212**, fluidically coupled to the fluid flow.

[0041] Heat recovery system **200** may further include one or more control valves **214**, one or more variable position valves **216**, one or more manifolds such as manifold **218** and manifold **220**, and expansion device **222**.

[0042] Control valves **214** may be actuated by a controller (not shown) to regulate HTF flow through heat recovery system **200**. As shown, a control valve may be positioned upstream from one of the heat exchangers **202**, upstream from heat storage device **100**, and/or at another position within heat recovery system **200** to regulate HTF flow. Depending on an operational state of the vehicle, one or more of the control valves may be actuated to regulate a temperature of the HTF. For example, when one or more control valves are closed, a volume of circulating HTF can be reduced such that the HTF can increase in temperature more rapidly.

[0043] Further, the HTF temperature may be regulated via actuation of variable control valve **216**. Such a control valve may be actuated to open at varying degrees to change a fluid flux of the HTF passing through variable control valve **216**. As shown, variable control valve **216** is positioned upstream from manifold **220**, and is included within bypass loop **224**. Bypass loop **224** may bypass heat storage device **100**. Therefore, bypass loop **224** may allow HTF to circulate without passing through heat storage device **100**. For example, to conserve heat stored in heat storage device **100**, variable control valve **216** may be adjusted to allow HTF fluid flow to flow through bypass loop **224**. In other words, bypass loop **224** may be a blending loop that blends cooler HTF fluid with warmer HTF fluid that circulates through heat exchanger **156**, heat storage device **100**, and/or one or more heat exchangers **202**. By blending HTF circulating through bypass loop **224** with other circulating HTF flow, an overall temperature of the circulating HTF may be reduced.

[0044] Further still, the HTF temperature may be regulated by routing all circulating HTF flow through bypass loop **226**. For example, bypass loop **226** may be an exhaust temperature boosting loop and bypass loop **226** may be a thermal recharging loop, depending on the operational state of engine **12** and/or the thermal capacity of heat storage device **100**. For example, bypass loop **226** may function as the exhaust temperature boosting loop when heat storage device **100** holds a thermal charge and the exhaust temperature is below a thresh-

old value. Further, heat exchanger **156** may be positioned upstream from one or more exhaust emissions control devices and heat storage device **100** may discharge heated HTF to be delivered to heat exchanger **156**. In this way, heated HTF may only be circulated through bypass loop **226** to increase a temperature of the exhaust flow, such that a time to reach catalyst light-off is reduced.

[0045] Further, as the thermal recharging loop, bypass loop **226** may extract heat from the exhaust flow to recharge heat storage device **100**. Thus, HTF may only flow through thermal charging loop **225** to increase the temperature of HTF via heat exchanger **156**. In this way, HTF may be heated by the exhaust flow to recharge the thermal capacity of heat storage device **100**. It may be advantageous to recharge heat storage device **100** in this way when a temperature of the heat storage device is below a threshold value. For example, after heat storage device has discharge its thermal capacity, and/or after the various engine systems are sufficiently warm.

[0046] In other words, one or more control valves **214** positioned upstream from heat exchangers **202** may be closed to reduce a volume of circulating HTF, and/or variable position control valve **216** may also be closed, such that circulating HTF only passes through bypass loop **226**, heat exchanger **156**, and heat storage device **100**. A method for regulating HTF flow through heat recovery system **200** by actuating one or more control valves is described with respect to FIG. 3.

[0047] As shown, manifolds **218** and **220** may be positioned in heat recovery system **200** where more than one pipe carrying HTF fluid merges. For example, manifold **218** may be configured to receive HTF fluid from one pipe and may include two HTF outlets. As another example, manifold **220** may be configured to receive HTF fluid from more than one pipe and may include more than one outlet. As shown, manifold **220** receives HTF flow from heat storage device **100** and from bypass loop **224**. Further, manifold **220** may have an outlet directed towards each heat exchanger **202** and/or to thermal charging loop **225**. It will be appreciated that manifolds **218** and **220** are provided as non-limiting examples, and thus, other configurations are possible without departing from the scope of this disclosure.

[0048] Expansion device **222** may be positioned downstream from the plurality of heat exchangers **202**. As shown, expansion device **222** is configured to receive HTF from each of the heat exchangers **202**, as well as thermal recharging loop **225**. For example, expansion device **222** may be provided for degassing. In other words, expansion device **222** may be positioned downstream from heat exchangers **202** and thermal recharging loop **225** to regulate a pressure of the incoming HTF flow.

[0049] It will be appreciated that heat recovery system **200** is provided by way of example, and thus, is not meant to be limiting. Therefore, it is to be understood that heat recovery system **200** may include additional and/or alternative features than those illustrated in FIG. 2 without departing from the scope of this disclosure. For example, the heat recovery system may include a three-way valve to regulate HTF flow to more than one engine system.

[0050] FIG. 3 schematically shows an example method **300** that may be used to operate heat recovery system **200**.

[0051] At **302**, method **300** includes determining if an engine has started. If the answer to **302** is NO, method **300** ends. If the answer to **302** is YES, method **300** continues to **304**.

[0052] At **304**, method **300** includes determining if an HTF temperature is below a threshold value. For example, the exhaust temperature may be an exhaust gas temperature upstream and/or downstream from an emissions control device. If the answer to **304** is YES, method **300** continues to **306**. If the answer to **304** is NO, method **300** continues to **308**.

[0053] At **306**, method **300** includes reducing a volume of circulating heat transfer fluid (HTF) and discharging a heat storage device (e.g., heat storage device **100**) to heat an exhaust system component, such as an exhaust system component. For example, reducing the volume may include closing one or more control valves to inhibit the circulating heat transfer fluid from being distributed to one or more engine systems. Further, discharging the heat storage device may include discharging stored thermal energy of the heat storage device, wherein the stored thermal energy is stored from a previous engine operation. Further still, the stored thermal energy may be transferred to the circulating heat transfer fluid and distributed to the exhaust system component. In some embodiments, the exhaust system component may be upstream from an emissions control device.

[0054] At **308**, method **300** includes distributing circulating HTF to one or more engine systems. For example, one or more control valves may be actuated to distribute circulating HTF to one or more of a cabin heating system, an engine coolant system, a transmission system, etc.

[0055] At **310**, method **300** includes determining if the one or more engine systems are sufficiently warm. If the answer to **310** is NO, method **300** returns to **306**. If the answer to **310** is YES, method **300** continues to **312**.

[0056] At **312**, method **300** includes recharging the heat storage device. For example, the volume of circulating HTF may be reduced and/or a bypass loop may be opened such that HTF is circulated through a heat exchanger thermally coupled to the exhaust system and through the heat storage device. In this way, a thermal capacity of the heat storage device may be increased.

[0057] At **314**, method **300** includes determining if a HTF temperature is above a threshold value. For example, the HTF may become too warm when the vehicle is in operation for an extended period of time. If the answer to **314** is NO, method **300** continues to **316**. If the answer to **314** is YES, method **300** continues to **318**.

[0058] At **316**, method **300** includes closing a blending loop. As such, the volume of circulating HTF is not adjusted in response to the temperature of the HTF.

[0059] At **318**, method **300** includes adjusting a variable position valve of the blending loop. As such, a dead volume of HTF is released from the blending loop to reduce the temperature of the circulating HTF. As described above, the temperature of the HTF may be regulated based on a position of the variable position valve. For example, the valve may be fully opened to rapidly cool the HTF. As another example, the valve may be partially opened to moderately cool the HTF.

[0060] It will be appreciated that method **300** is provided by way of example, and thus, is not meant to be limiting. Therefore, it is to be understood that method **300** may include additional and/or alternative steps than those illustrated in FIG. 3 without departing from the scope of this disclosure. Further, it will be appreciated that method **300** is not limited to the order illustrated; rather, one or more steps may be rearranged or omitted without departing from the scope of this disclosure. For example, one or more portions of method

300 may occur without starting the engine. As described above, the heat storage device may be activated to discharge without operating the engine.

[0061] Various conduits may be referred to as pipes, which can encompass various forms of conduits, passages, connections, etc., and are not limited to any specific cross-sectional geometry, material, length, etc.

[0062] It will be appreciated that the configurations and routines disclosed herein are exemplary in nature, and that these specific embodiments are not to be considered in a limiting sense, because numerous variations are possible. For example, the above technology can be applied to V-6, I-4, I-6, V-12, opposed 4, and other engine types. The subject matter of the present disclosure includes all novel and non-obvious combinations and sub-combinations of the various systems and configurations, and other features, functions, and/or properties disclosed herein.

[0063] The following claims particularly point out certain combinations and sub-combinations regarded as novel and non-obvious. These claims may refer to “an” element or “a first” element or the equivalent thereof. Such claims should be understood to include incorporation of one or more such elements, neither requiring nor excluding two or more such elements. Other combinations and sub-combinations of the disclosed features, functions, elements, and/or properties may be claimed through amendment of the present claims or through presentation of new claims in this or a related application. Such claims, whether broader, narrower, equal, or different in scope to the original claims, also are regarded as included within the subject matter of the present disclosure.

1. A heat storage device for an engine, comprising:
 - an inlet passage thermally coupling the heat storage device to an exhaust passage;
 - a stack of phase changing materials positioned radially about a central feed passage, the stack positioned within an interior of a double wall vessel; and
 - an outlet passage thermally coupling the heat storage device to another engine system.
2. The device of claim 1, wherein the inlet passage is coupled to an end of the heat storage device at a centrally located position.
3. The device of claim 1, wherein the outlet passage is coupled to another opposite end of the heat storage device, the outlet passage at a top periphery located position.
4. The device of claim 1, further comprising a vacuum passage in fluidic communication with a vacuum space of the double wall vessel.
5. The device of claim 4, wherein the double wall includes outer walls and inner walls, and wherein a vacuum jacket is positioned between the outer walls and the inner walls.
6. The device of claim 5, wherein the inlet passage and the outlet passage include outer walls, inner walls, and a vacuum space between the outer walls and the inner walls, the vacuum space in fluidic communication with the vacuum jacket.
7. A system for an engine, comprising:
 - a heat exchanger in fluidic communication with an exhaust passage;
 - a heat storage device including a phase changing material;
 - a first pipe thermally coupling the heat exchanger to the heat storage device; and
 - a second pipe thermally coupling the heat storage device to another engine system.
8. The system of claim 7, wherein the first and second heat pipes include a phase changing material.

9. The system of claim 7, wherein the phase changing material of the heat storage device is positioned radially about a central feed passage.

10. The system of claim 7, wherein the heat storage device includes more than one phase changing material, each phase changing material having a different phase transfer temperature.

11. The system of claim 7, wherein the heat storage device is a double walled vessel including an inner vessel and an outer vessel separated by a vacuum jacket.

12. The system of claim 11, further comprising a plurality of anti-radiation foils positioned within the vacuum jacket.

13. The system of claim 12, further comprising a vacuum passage fluidically coupling the vacuum jacket to a vacuum pump.

14. The system of claim 7, wherein the heat storage device is tilted 5 degrees from a horizontal, wherein the another system is separate from the heat exchanger and the heat storage device, wherein only the second heat pipe couples the another system to the heat storage device, wherein the another system includes a fluid conduit separate from the second heat pipe, wherein the fluid conduit is in thermal contact with the second heat pipe at a position coinciding with another heat exchanger, wherein the another heat exchanger is a location where heat transfer occurs between the second heat pipe and the fluid conduit, and wherein the another system is selected

from the group consisting of a cabin heating system, a coolant system, and a transmission system.

15. A heat transfer system for an engine, comprising:
a heat exchanger coupled to an exhaust passage downstream from a catalytic converter;
a heat storage device, coupled to the heat exchanger via an inlet, including a cylindrical stack positioned between two retention plates; and
an outlet coupling the heat storage device to another engine system.

16. The heat transfer system of claim 15, wherein the heat storage device is a double walled vessel including inner walls, outer walls, and a vacuum jacket between the inner and outer walls.

17. The heat transfer system of claim 16, wherein the stack circumferentially surrounds a central feed passage that supplies a heat transfer fluid to the heat storage device.

18. The heat transfer system of claim 17, further comprising one or more springs coupled to one retention plate and the inner walls of the double walled vessel.

19. The heat transfer system of claim 16, wherein the inlet and the outlet include double walls with a vacuum space between inner and outer walls, and wherein the vacuum space is continuous with the vacuum jacket.

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