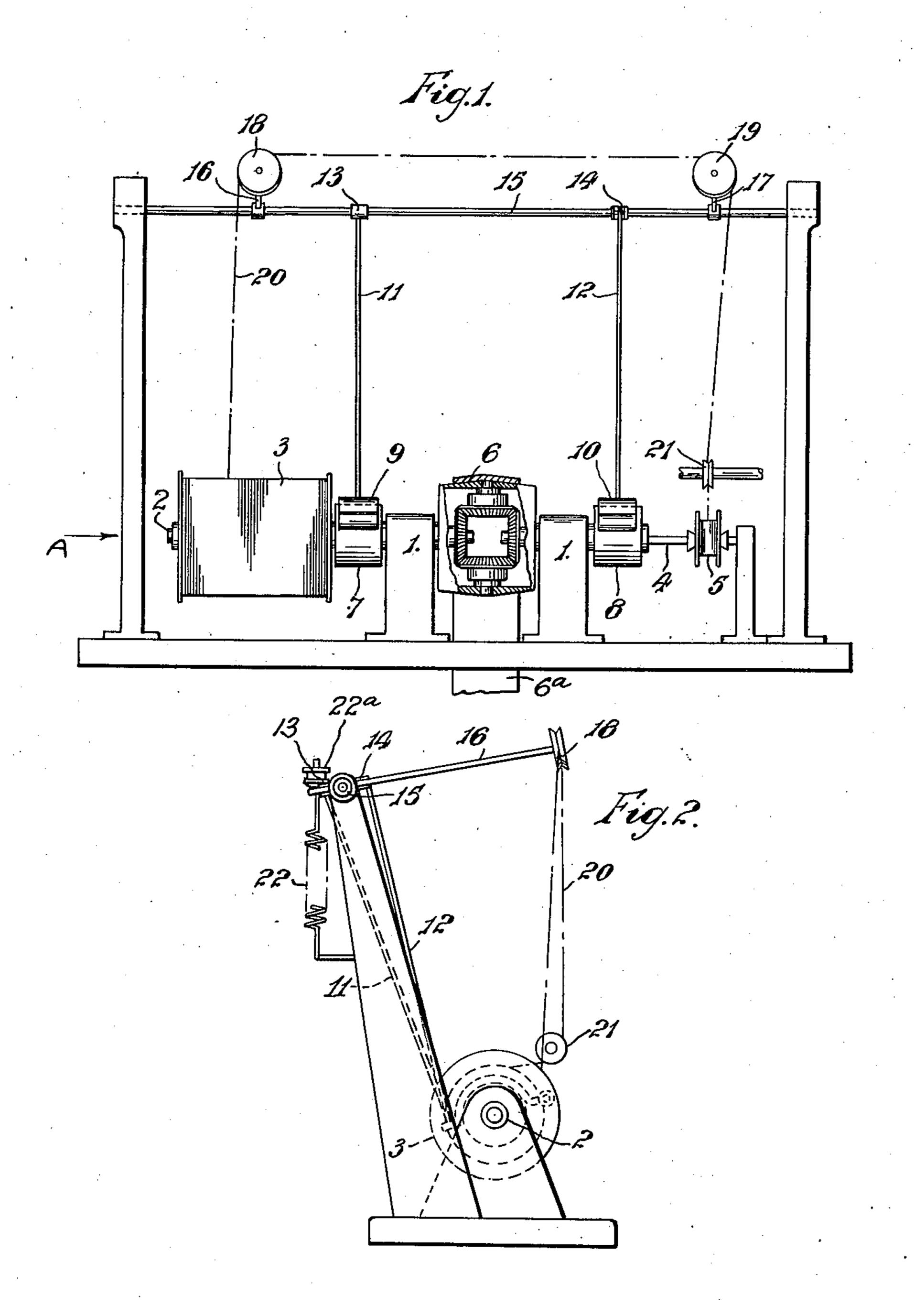
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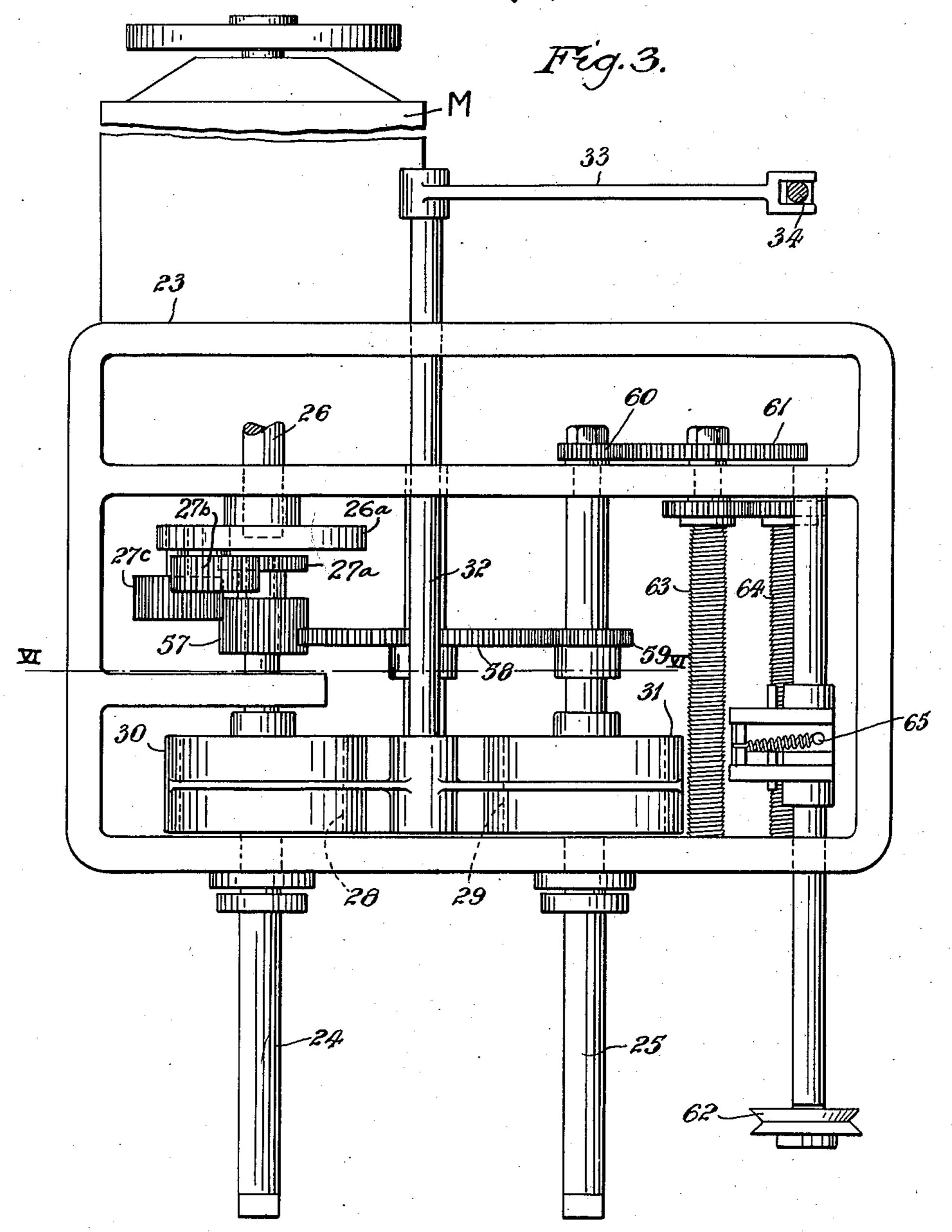
E.L. WILDY.

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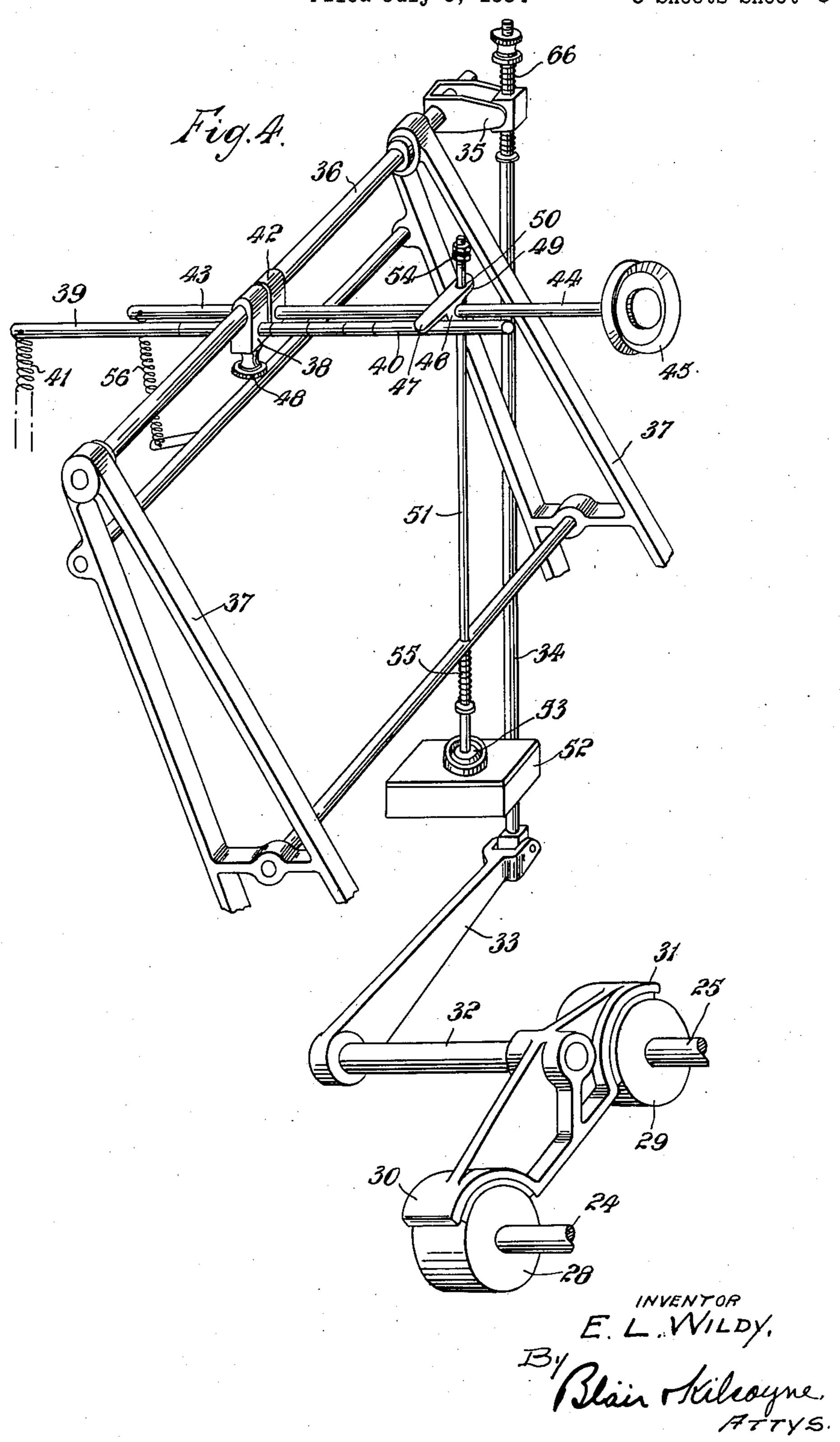
EL. WILDY.

By Blair Thileryne

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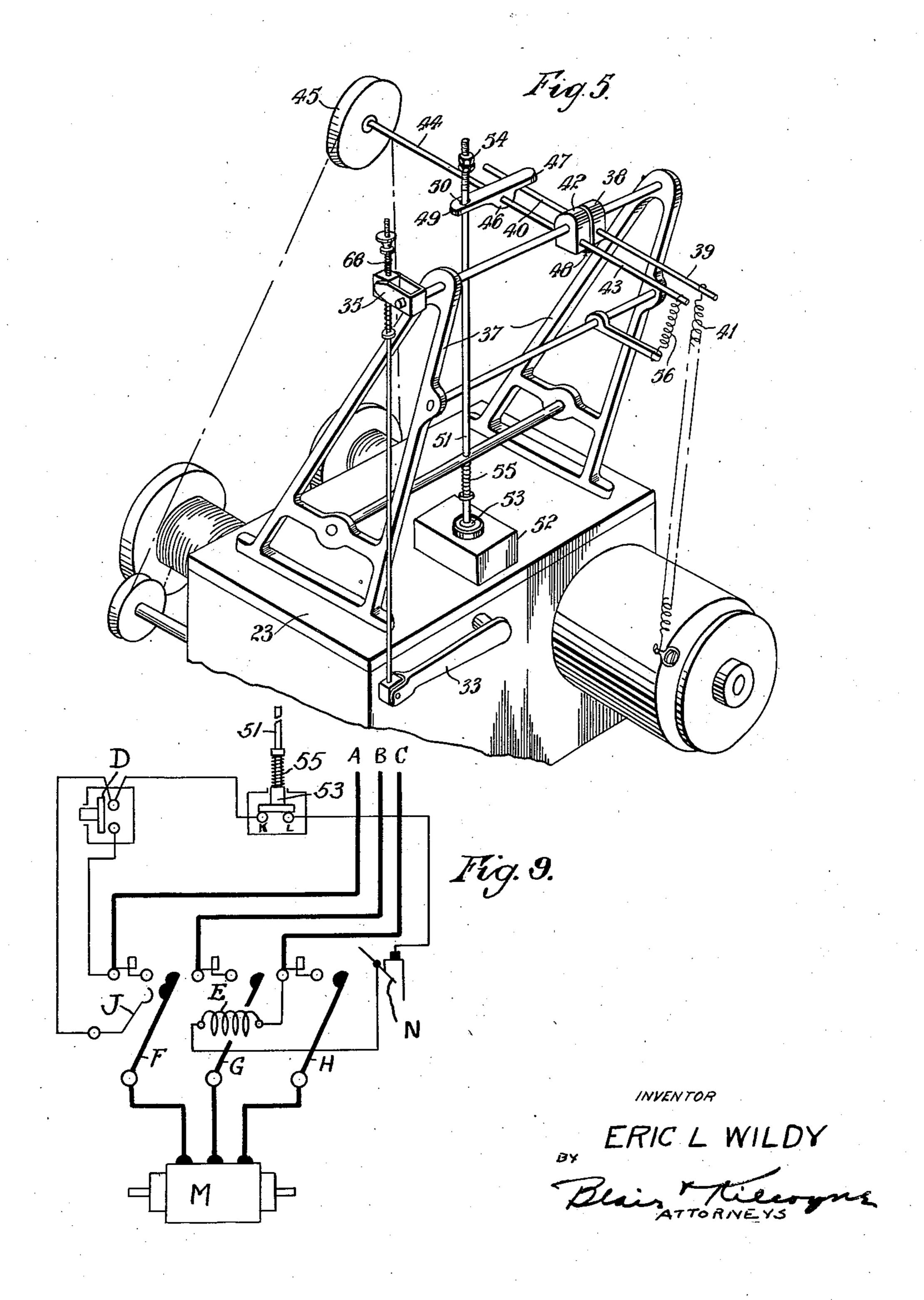
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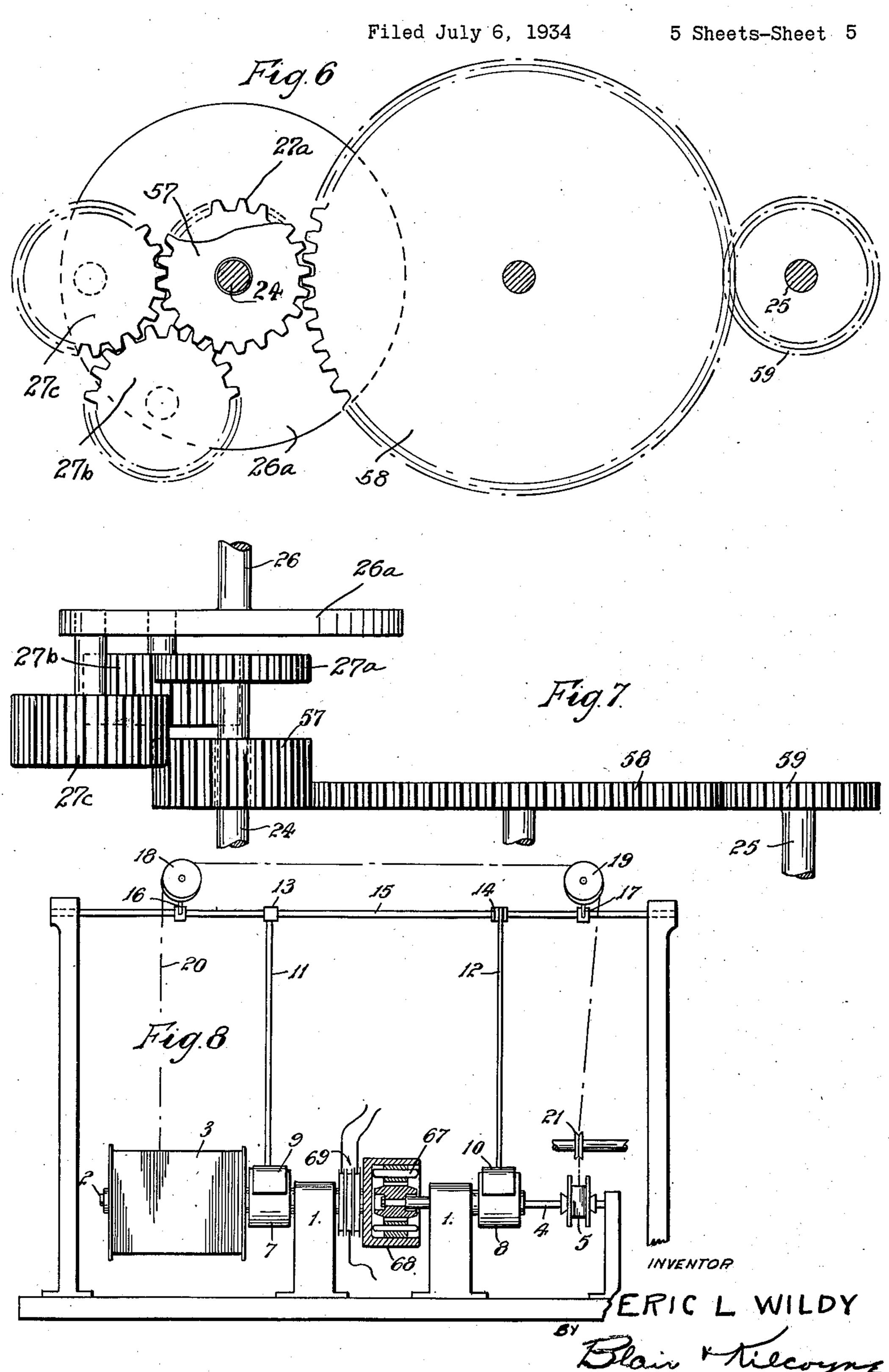
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UNITED STATES PATENT OFFICE

2,012,208

WINDING MACHINE FOR FILAMENTS

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12 Claims. (Cl. 242-45)

This invention relates to machines for winding of rotation of one or other of the bobbin spindles as desired comprises two interconnected brakes, winding wire.

When winding wire from one bobbin to another, 5 the external diameters formed by the winding on the bobbins vary as the amount of wire on the two bobbins increases and decreases respectively. It is necessary therefore to control the relative speeds of rotation of the bobbins in such a way that the speed of rotation of the bobbin having the smaller external diameter of winding is correspondingly greater than that of the bobbin with the larger external diameter of winding, and that the proper relation between these speeds is constantly maintained in order, on the one hand, that there shall be no slackness in the wire, and on the other hand that the tension in the wire shall not become excessive but shall remain as uniform as possible.

The object of the invention is to provide a machine for winding filaments in which the relative speeds of rotation of the bobbins are continuously and automatically controlled.

The term "bobbin" used throughout this specification is intended to be interpreted in its broadest sense to include formers, reels, etc.

According to the invention, a machine for winding filaments comprises two spindles for the reception of unwinding and winding bobbins respectively connected with each other through a differential mechanism in association with means actuated by the tension of the filament being wound from one bobbin to another for retarding the speed of rotation of the spindles in such a manner that the speeds of rotation of the two spindles are continuously controlled.

The machine may be driven in any convenient manner; for example, the differential mechanism may be housed within a driving pulley, or a driving shaft may drive the two spindles carrying the bobbins through the differential mechanism or any other convenient type of drive may be used.

Although the more general mechanical forms of differential mechanism are most commonly utilized according to the present invention, other forms of differential mechanism will also serve. For example, an alternating current field motor with its "rotor" attached to a spindle for the reception of one bobbin, and its "stator" attached 50 to a spindle for the reception of another bobbin may be used. Both the "rotor" and "stator" of the motor are adapted and arranged in such a manner that they are both free to rotate. The electrical current to the "stator" may be supplied through the agency of suitable slip ring and brush gear. The braking mechanism associated with the two spindles is similar in all respects to that utilized in connection with the embodiments hereinafter described.

Preferably the means for retarding the speed

as desired comprises two interconnected brakes, one to operate on each bobbin spindle and arranged in such a manner that, as the external diameter of winding on the unwinding bobbin decreases and the external diameter of the winding on the winding bobbin increases, the tension of the wire being wound from one bobbin to another tends to increase, that is to say, the bobbin which is winding the wire tends to run at too great 10 a speed, one of said brakes is applied to the spindle carrying the winding bobbin whilst the other brake is relieved, with the result that the speed of rotation of the spindle carrying the winding bobbin is reduced and simultaneously through the ac- 15 tion of the differential mechanism the speed of rotation of the spindle carrying the unwinding bobbin is increased.

Two embodiments of an apparatus embodying the invention are hereinafter described and 20 shown in the accompanying drawings, wherein:—

Fig. 1 is a diagrammatic front elevation,

Fig. 2 is a side elevation looking in the direction of the arrow "A",

Fig. 3 is a plan view of a further example clear- 25 ly showing the differential mechanism together with part of the brake mechanism, and

Fig. 4 is a perspective view clearly showing the operation of the brake mechanism.

Fig. 5 is a perspective view of the complete 30 machine from behind,

Fig. 6 is a sectional view taken along line VI—VI of Fig. 3, clearly showing the relative disposition of certain parts of the differential mechanism,

Fig. 7 is a plan view of certain parts as shown in Fig. 6,

Fig. 8 is a view similar to Fig. 1 but which incorporates an electrical alternating current field motor.

Fig. 9 shows the electrical circuit of the motor which drives certain portions of the mechanism.

Referring first to Figs. 1 and 2 of the drawings:—The frame I supports a shaft 2 which carries an unwinding bobbin 3 and a shaft 4 45 carrying a winding bobbin 5, which is shown in the drawings as a former for a coil winding. The shafts 2 and 4 are connected with each other through a differential mechanism 6. The machine is driven through the agency of a belt 6a. 50

On the shafts 2 and 4 are two brake drums 7 and 8 which are acted on by brakes 9 and 10, which are connected to and operate through rods 11 and 12 to levers 13 and 14 mounted on a fulcrum rod 15 which is supported by the framework 1 (see Fig. 2). The rod 12 is attached to the lever 14 which is mounted in front of the fulcrum rod 15 and the rod 11 is attached to the lever 13 which is mounted behind the fulcrum rod 15.

Rigidly connected to the fulcrum rod 15 are two arms 16 and 17 which are adjustable with regard to length, having at their extremities two pulley wheels 18 and 19. The wire 20 (shown 5 in dot and pick lines) from the unwinding bobbin 3 passes over the pulley wheels 18 and 19 over a spreader member 21 and on to the winding bobbin 5.

It will be seen with reference to Fig. 2 that 10 if the arms 16 and 17 move upwardly, the brake 9 is applied to the drum 7 and the arrangement is such that the brake 10 is relieved. In the same manner, if the arms 16 and 17 are moved in a downward direction, the brake 10 is applied to the drum 8 and the brake 9 is relieved.

The operation of the machine is as follows:— The wire 20 from the unwinding bobbin 3 is passed over the pulley wheels 18 and 19 and the spreader 21, and its free end fixed to the wind-20 ing bobbin 5. The wire 20 being slack, the arms 16 and 17 have moved upwardly under the action of a spring 22 (see Fig. 2) adjustable by a knurled nut 22a so that the brake 9 is applied to the drum 7. The tension spring is initially 25 adjusted to suit the type of wire being wound and may be adjusted to suit the weight of the respective bobbins.

Motion is imparted to the differential mechanism 6 by a belt 6a and tends to rotate the shafts 30 2 and 4. The shaft 4 rotates freely but the brake mechanism 9, 7, is tending to retard the rotation of the shaft 2. As the shaft 4 rotates the tension in the wire 20 increases to such an extent that the arms 16 and 17 are drawn downwardly, relieving the pressure of the brake 9 on the drum 7.

As the external diameter of the winding on the bobbin 5 is less than that on the bobbin 3, the wire would tend to become slack owing to the bobbin 3 unwinding the wire 20 at too great a speed. This tendency is corrected by the arms 16 and 17 rising with any decrease in wire tension, causing the brake mechanism 9, 7, to retard the speed of rotation of the spindle 2 and. simultaneously through the action of the differential mechanism 6 the speed of rotation of the shaft 4 is increased.

As the external diameter of the winding on the bobbin 5 increases, with a consequent tendency for the wire tension to increase, the arms 16 and 17 move slightly in a downward direction, and the brake mechanism 10, 8, retards the motion of the shaft 4, and simultaneously through the action of the differential mechanism 6 the speed of rotation of the shaft 2 carrying the unwinding bobbin 3 is increased.

Referring now to Figs. 3 and 4 of the drawings, a casing 23 carries a spindle 24 adapted to receive an unwinding bobbin (not shown), and a spindle 25 adapted to receive a winding bobbin (not shown). The spindles 24 and 25 are connected together to a driving shaft 26 through a spur-gear differential mechanism constituted by the plate 26a, the wheels 27a, 27b, 27c, 57, 5865 and 59. The operation of this differential mechanism is hereinafter more fully described. The shaft 26 is driven by a motor "M".

On the spindles 24 and 25 respectively within the casing 23 are two brake drums 28 and 29, which are arranged to be acted upon by brake shoes 30 and 31. As will readily be seen from the drawings, the brake shoes 30 and 31 are of unit construction and are rigidly mounted upon one end of a shaft 32 so that a partial rotation of the shaft 32 applies either the brake shoe 30

on the brake drum 28 or the brake shoe 31 on the brake drum 29, depending upon the direction of rotation of the spindle 32.

One end of an arm 33 is pivotally attached to the other end of the shaft 32 and is arranged in such a manner that an upward movement of the arm 33 causes the brake shoe 30 to act upon the brake drum 28, whilst a downward movement causes the brake shoe 31 to act upon the brake drum 29.

Pivotally attached to the other end of the arm 33 is the lower end of a rod 34, the upper end of which is pivotally attached to one end of a short arm 35, the other end of which is rigidly mounted on a fulcrum rod 36. The fulcrum rod 36 is car- 15 ried on a frame work 37 which is held positioned on the lid of the casing 23 (see more particularly Fig. 5).

Rigidly mounted on the fulcrum rod 36 is a block 38 in which a lever 39, 40, is held, a move- 20 ment of which either upwardly or downwardly partially rotates the fulcrum rod 36, which in turn moves the rod 34 and the lever 33 to operate the brake mechanism in the manner hereinbefore described.

A spring 41 normally tends to pull the arm 40 of the lever 39, 40, upwardly; that is to say, to the position in which the brake shoe **30** is applied to the brake drum 28, which will tend to retard the rotation of the spindle 24.

Loosely mounted on the fulcrum rod 36 is a block 42 which carries a lever 43, 44, the arm 44 of which is provided at its end with a pulley 45, over which the wire being wound from one bobbin to another passes.

The depending limb 46 of a T-shaped member is fixed on the arm 44 of the lever 43, 44, one arm 47 of the T-shaped member being adapted to engage the arm 40 of the lever 39, 40, when the arm 44 is pulled or moved in a downward direction 40 to operate the brake mechanism.

The effective length of the arm 40 of the lever 39, 40, and consequently the point of engagement of the arm 47 of the T-shaped member, can be adjusted by sliding the lever in the block 38, screw 45 adjusting means 48 being provided for this purpose.

The other arm 49 of the T-shaped member is provided with an aperture **50** so that it can ride loosely over a rod 51.

The circuit of an electric motor which imparts motion to the driving shaft 26 is connected through a stop control mechanism 52 which operates in such a manner that when a button 53 is momentarily raised, the circuit to the motor 55 is broken and thus the machine is stopped.

This circuit (see Fig. 9) operates as follows: Three-phase A. C. current is supplied from the mains A, B, C, as indicated.

On depressing the push-button of the starting 60 switch D the circuit A, C, is completed, the solenoid E is energized and the 3-pole switch, F, G, H, is closed by the action of this solenoid.

In closing the arm F of the 3-pole switch also makes contact with a secondary switch J, which 65 serves to keep the solenoid circuit closed when pressure is removed from the starting button D, so that pressure on the switch D need only be momentary. The motor M will now be in operation by virtue of current from the three-phase 70 A. C. main, and will continue to run until the breaking of the wire causes the arm 51 to lift the button 53 momentarily against the action of the spring 55, thus breaking the solenoid circuit at the points K and L, when the solenoid becomes de- 75

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energized and allows the switch arms F, G and H to return to the position shown in the drawings. Overloading of the motor circuit is guarded against by the inclusion of a thermal switch of s any suitable type, as indicated at N.

The specific circuit herein described forms no part of the instant invention except in so far as it enters into the combination shown, and is merely illustrative of a suitable circuit for use

10 with this apparatus.

The lower end of the rod 51 is attached to the button 53, and its upper end is provided with an

adjustable abutment 54.

When the arm 49 of the T-shaped member rises and strikes the abutment 54 the rod 51 rises, thus lifting the button 53 to break the circuit. When the arm 49 of the T-shaped member is moved away from the abutment 54 a spring 55 returns the rod 51 to its normal position.

A spring 56 is arranged so as to cause the arm 44 of the lever 43, 44, and thus the arm 49 of the T-shaped member to strike the abutment 54 immediately the load on the pulley 45 is relieved, that is to say, whenever for any reason there is no

25 wire passing over the pulley 45.

The spindles 24 and 25 are connected to each other through the gears 27a, 27b, 27c, 51, 58, 59, constituting part of the differential mechanism 27. The spreader mechanism of any suitable known construction is connected to the spindle 25 through gears 60, 61. Briefly, the spreader mechanism is of a known type, comprising two threaded members 63 and 64 upon which a carriage member 65 travels. The ar-35 rangement is such that when the carriage member 65 reaches the end of its travel on either of the threaded members 63, 64, it is automatically returned on the thread of the other threaded member in any suitable conventional manner. It will be seen from the drawings that the speed of the winding spindle 25 and in addition the spreader mechanism maintains a constant pitch. It will be understood that the spreader mechanism may be associated with any of the necessary devices utilized in connection with coil winding. The spreader mechanism is provided with a pulley 62 over which the wire which is being wound passes.

A spring means 66 serves for adjusting the

brake mechanism.

As hereinbefore stated, the spindles 24 and 25 are connected together to a driving shaft 26 which is itself driven by means of the motor "M" through a spur-gear differential mechanism 27. This differential mechanism 27 is illustrated diagrammatically in Figs. 6 and 7 and comprises wheels "A", "B", "C", "D" and "E", the wheel "E" representing the wheel 58 and the wheel "B" representing the wheel 57 as shown in Fig. 4 of the drawings.

It will be seen from Figs. 6 and 7 that the wheel "A" is formed integrally with the spindle 24, the wheel "B" is loosely mounted thereon so as to be capable of rotation independently of the spindle 24. This wheel is also in mesh with the wheel "E". The wheel "B" is in mesh with the wheel "C" which itself is in mesh with the wheel "D" which in turn is in mesh with the wheel "A". The wheels "C" and "D" are carried by the circular plate "F" but are capable of rotating freely about their own axes.

The operation of the differential mechanism is as follows:—

Motion is transmitted by the shaft 26 causing the circular plate "F" to rotate.

Should there be no force tending to retard the speed of rotation of either of the spindles 24 and 25 then the shaft 26 will transmit its motion equally to the wheel "A", that is, to the spindle 24, to the wheel "E", the wheel 59, and 5

consequently the spindle 25.

Now when the shaft 26 rotates it carries the circular plate 26a and with it the wheels 27b and 27c. The wheel 27b being in mesh with the wheel 27a causes the wheel 27a to rotate with 10 the same speed of rotation, that is to say, with the same speed of rotation as the shaft 26, since wheels 27b and 27c do not rotate independently on their own axes but rotate with the circular plate 26a. The wheel 27c being in mesh with 15 the wheel 57 causes the wheel 57 to rotate and through the wheel 58, the wheel 59, that is to say, the spindle 25, and the arrangement in the size of gears is such that the speed of rotation of the wheel 59 is the same as the 20 wheel 27a.

Now should the spindle 24 be held completely against rotation, the wheel "D" rotating on its own axis will roll around the wheel "A" but the wheel "C" will transmit its motion through the 25 wheel "B", which is free to rotate on the now stationary spindle 24, through the wheel "E" to the wheel 59, resulting in this wheel rotating at

twice the speed of the shaft 26.

Should the spindle 25 be held against rotation, 30 then the wheels 59, "E" and "B" will remain stationary. The wheel "C" now rotates on its own axis and merely rolls around the wheel "B" but the wheel "D", also rotating on its own axis and meshing with the wheel "A", causes it, and there- 35 fore the spindle 24, to rotate at twice the speed of rotation of the shaft 26. It will be seen, therefore, that when only a partial retardation is effected, in the speed of rotation of one of the spindles 24 and 25, through the action of the 40differential mechanism there is an increase in the speed of rotation of the other spindle—that is to say, the spindle which is not retarded.

The operation of the machine is as follows:— The wire from the unwinding bobbin which is 45 on the spindle 24 is passed over the pulley wheel 45, the spreader pulley 62 and on to the winding bobbin which is situated on the spindle 25, where its free end is fixed. The wire being slack, no movement has taken place of the lever 50 43, 44, so that the brake shoe 30 is still acting upon the brake drum 28, and so tending to retard the speed of rotation of the spindle 24. As soon as the shaft 26 commences to rotate the spindle 24 is held momentarily by the action of 55 the brake shoe 30 on the brake drum 28 against motion, but the spindle 25 commences to rotate, gradually increasing the tension in the wire, causing the pulley 45 to be pulled downwardly so that the arm 47 of the T-shaped member en- 60 gages with the arm 40 of the lever 39, 40, and pulls it slightly downwardly, which also through the agency of the rod 34 moves the arm 33 downwardly, thus partially relieving the pressure of the brake shoe 30 on the brake drum 28, so that 65 the spindle 24 commences to rotate.

Now should the tension in the wire tend to increase during the winding from one bobbin to another, the pulley 45 will be pulled downwardly, causing the brake shoe 31 to act upon the brake 70 drum 29 to retard the speed of rotation of the spindle 25 when through the action of the differential mechanism 27 the speed of rotation of the spindle 24 increases.

Should the tension in the wire tend to de- 75

crease, then the pulley 45 rises under the action of the spring 56 causing the arm 33 to be moved upwardly and through the agency of the brake mechanism the speed of rotation of the spindle 5 24 is retarded when through the action of the differential mechanism 27 the speed of rotation of the spindle 25 increases.

When a winding operation is completed, that is to say, when there is no more wire on the un10 winding bobbin, or should the wire break during winding, the pulley 45 relieved of its load moves upwardly under the action of the spring 56 and the arm 49 of the T-shaped member strikes the abutment 54, causing the electric circuit to be 15 broken and stopping the driving motor.

Fig. 8 shows diagrammatically an arrangement similar to Figs. 1 and 2 but in which the differential mechanism 6 shown in Fig. 1 is replaced by an electric alternating current field motor, the "rotor" 67 of which is formed integrally with the shaft 4 which carries the winding bobbin 5. The "stator" 68 is formed integrally with the shaft 2 which carries the unwinding bobbin 3. Electric current is supplied to the motor constituted by the "rotor" 67 and "stator" 68 through the agency of suitable slip-ring and brush-gear 69.

The operation of the machine is similar in all other respects to that shown in Figs. 1 and 2.

It will be seen that the winding machine for wire or similar material according to the invention so continuously adjusts the relative speeds of rotation of the unwinding and winding bobbins respectively that the tension in the wire which is being wound is kept as uniform as possible, and in starting or stopping the machine the speed of rotation of the bobbins changes relatively by virtue of the differential mechanism associated with the hereinbefore described means for retarding the speed of rotation of either of the spindles carrying the bobbins, thus preventing any overrun or tightening of the wire being wound.

It will be understood that the winding machine for wire herein-described is also applicable to textile yarns and filaments of all descriptions. What is claimed is:—

1. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound, and the other on which filament is to be wound, a differential mechanism, driving means for said spindles, said spindles being connected with each other and to the driving means through said differential mechanism, of a brake cooperating with one of said spindles, a second brake cooperating with the other spindle, means actuated by the tension of the filament which is being wound from one of said bobbins to the other, said means being adapted to control both the two said brakes so as to apply the one whilst the other is relieved.

In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound, and the other on which filament is to be wound, a differential mechanism, driving means for said spindles, said spindles being connected with each other and to the driving means through said differential mechanism, of brake drums and interconnected brake shoes, said brake drums being provided one on each of the said two spindles, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, one of said brake shoes being arranged to be applied to one of said drums
 for retarding the speed of rotation of one of the

said spindles, whilst the other brake is relieved, allowing the speed of rotation of the other of the said spindles to accelerate.

3. In a machine for winding filaments, the combination with two spindles for the reception of 5 bobbins one from which filament is to be unwound and the other on which filament is to be wound, a differential mechanism, driving means for said spindles, said spindles being connected with each other, and to the driving means, 10 through said differential mechanism, of brake drums and interconnected brake shoes, said brake drums being provided one on each of said spindles, a pulley over which the filament which is being wound from one bobbin to another passes, 15 said pulley being connected to and operating said interconnected brake shoes, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, a downward movement of the said pulley causing one of 20 said brake shoes to be applied to one of said brake drums to retard the speed of rotation of one of the spindles and an upward movement of said pulley causing the other of said brake shoes to be applied to the other of said brake drums to re- 25 tard the speed of rotation of the other of said spindles.

4. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical alternating current field motor, one of said spindles being connected to the "stator" and the other of said spindles to the "rotor", of a brake 35 cooperating with one of said spindles, a second brake cooperating with the other spindle, means actuated by the tension of the filament which is being wound from one of said bobbins to the other, said means being adapted to control both 40 the two said brakes so as to apply the one whilst the other is relieved.

5. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be un- 45 wound, and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical alternating current field motor, one of said spindles being connected to the "stator" and the other of said spindles to the "rotor", of brake 50 drums and interconnected brake shoes, said brake drums being provided one on each of the said two spindles, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, one of said brake 55 shoes being arranged to be applied to one of said drums for retarding the speed of rotation of one of the said spindles, whilst the other brake is relieved, allowing the speed of rotation of the other of the said spindles to accelerate.

6. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical 65 alternating current field motor, one of said spindles being connected to the "stator" and the other of said spindles to the "rotor", of brake drums and interconnected brake shoes, said brake drums being provided one on each of said 70 spindles, a pulley over which the filament which is being wound from one bobbin to another passes, said pulley being connected to and operating said interconnected brake shoes, said brake shoes being actuated by the tension of the fila-

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ment which is being wound from one bobbin to another, a downward movement of the said pulley causing one of said brake shoes to be applied to one of said brake drums to retard the speed of rotation of one of the spindles and an upward movement of said pulley causing the other of said brake shoes to be applied to the other of said brake drums to retard the speed of rotation of the other of said spindles.

7. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound, and the other on which filament is to be wound, a differential mechanism, driving means 15 for said spindles, said spindles being connected with each other and to the driving means through said differential mechanism, of a brake cooperating with one of said spindles, a second brake cooperating with the other spindle, means actuated by the tension of the filament which is being wound from one of said bobbins to the other, said means being adapted to control both the two said brakes so as to apply the one whilst the other is relieved, and means also actuated by the ten-25 sion of the filament which is being wound from one of said bobbins to another for stopping the machine.

8. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound, and the other on which filament is to be wound, a differential mechanism, driving means for said spindles, said spindles being connected with each other and to the driving means through 35 said differential mechanism, of brake drums and interconnected brake shoes, said brake drums being provided one on each of the said two spindles, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, one of said brake shoes being arranged to be applied to one of said drums for retarding the speed of rotation of one of the said spindles, whilst the other brake is relieved allowing the speed of rotation of the other of the said spindles to accelerate, and means also actuated by the tension of the filament which is being wound from one of said bobbins to another for stopping the machine.

9. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be unwound and the other on which filament is to be wound, a differential mechanism, driving means for said spindles, said spindles being connected with each other, and to the driving means, through said differential mechanism, of brake drums and interconnected brake shoes, said brake drums being provided one on each of said spindles, a pulley over which the filament which is being wound from one bobbin to another passes, said pulley being connected to and operating said interconnected brake shoes, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, a downward movement of the said pulley causing one of said brake shoes to be applied to one of said brake drums to retard the speed of rotation of one of the spindles and an upward movement of said pulley causing the 70 other of said brake shoes to be applied to the other of said brake drums to retard the speed of

rotation of the other of said spindles, and means also actuated by the tension of the filament which is being wound from one of said bobbins to another for stopping the machine.

10. In a machine for winding filaments, the 5 combination with two spindles for the reception of bobbins, one from which filament is to be unwound and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical alternating current field motor, one of said spin- 10 dles being connected to the "stator" and the other of said spindles to the "rotor", of a brake cooperating with one of said spindles, a second brake cooperating with the other spindle, means actuated by the tension of the filament which is be- 15 ing wound from one of said bobbins to the other, said means being adapted to control both the two said brakes so as to apply the one whilst the other is relieved, and means also actuated by the tension of the filament which is being wound 20 from one of said bobbins to another for stopping the machine.

11. In a machine for winding filaments, the combination with two spindles for the reception of bobbins, one from which filament is to be 25 unwound, and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical alternating current field motor, one of said spindles being connected to the "stator" and the other of said spindles to the "rotor", of brake 30 drums and interconnected brake shoes, said brake drums being provided one on each of the said two spindles, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, one of said 35 brake shoes being arranged to be applied to one of said drums for retarding the speed of rotation of one of the said spindles, whilst the other brake is relieved, allowing the speed of rotation of the other of the said spindles to accelerate, and means also actuated by the tension of the filament which is being wound from one of said bobbins to another for stopping the machine.

12. In a machine for winding filaments, the combination with two spindles for the reception 45 of bobbins, one from which filament is to be unwound and the other on which filament is to be wound, a "stator" and a "rotor" of an electrical alternating current field motor, one of said spindles being connected to the "stator" and the 50other of said spindles to the "rotor", of brake drums and interconnected brake shoes, said brake drums being provided one on each of said spindles, a pulley over which the filament which is being wound from one bobbin to another 55passes, said pulley being connected to and operating said interconnected brake shoes, said brake shoes being actuated by the tension of the filament which is being wound from one bobbin to another, a downward movement of the said pulley causing one of said brake shoes to be applied to one of said brake drums to retard the speed of rotation of one of the spindles and an upward movement of said pulley causing the other of said brake shoes to be applied to the other of said brake drums to retard the speed of rotation of the other of said spindles, and means also actuated by the tension of the filament which is being wound from one of said bobbins to another for stopping the machine. ERIC L. WILDY.