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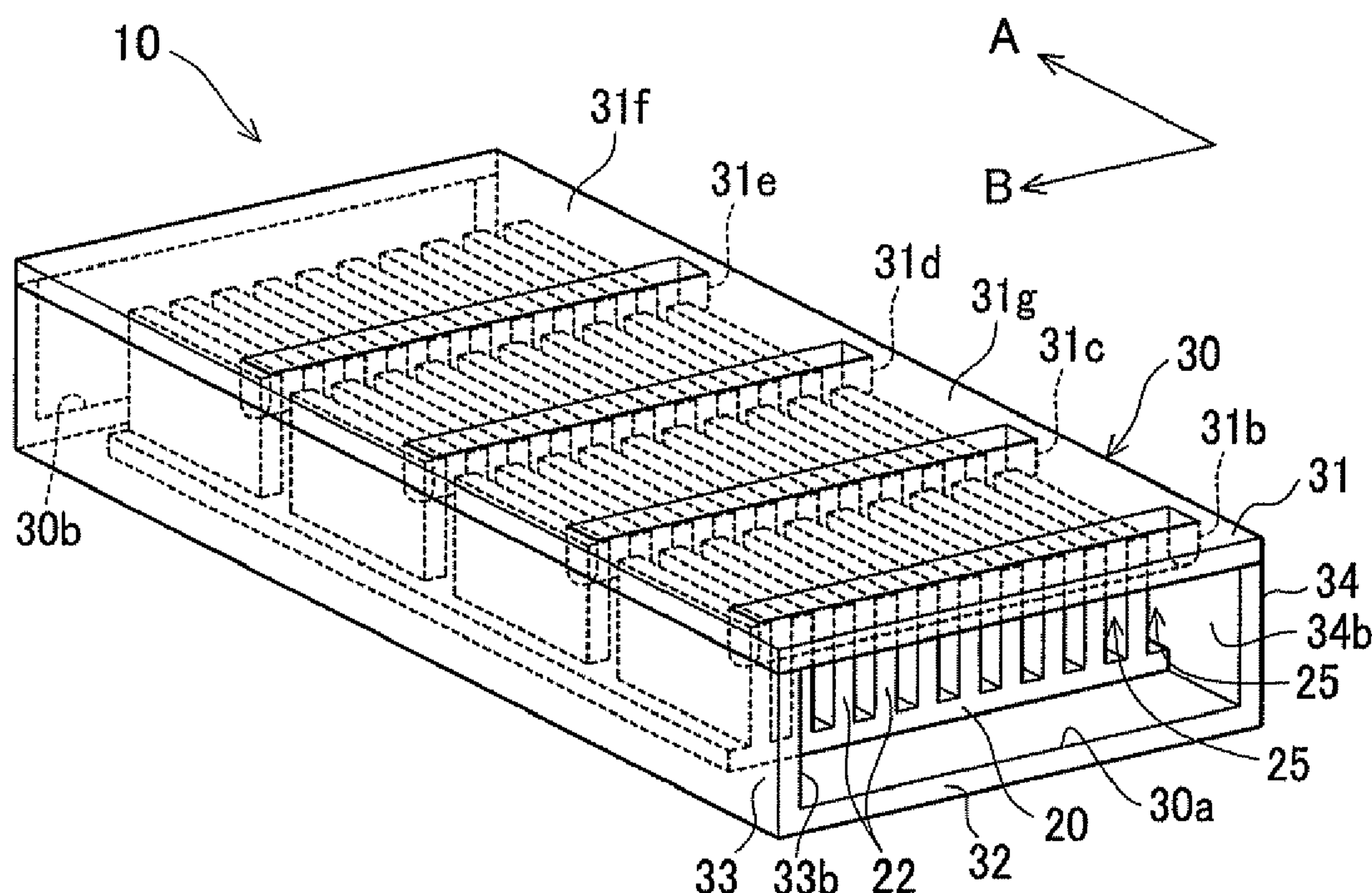


FIG.1

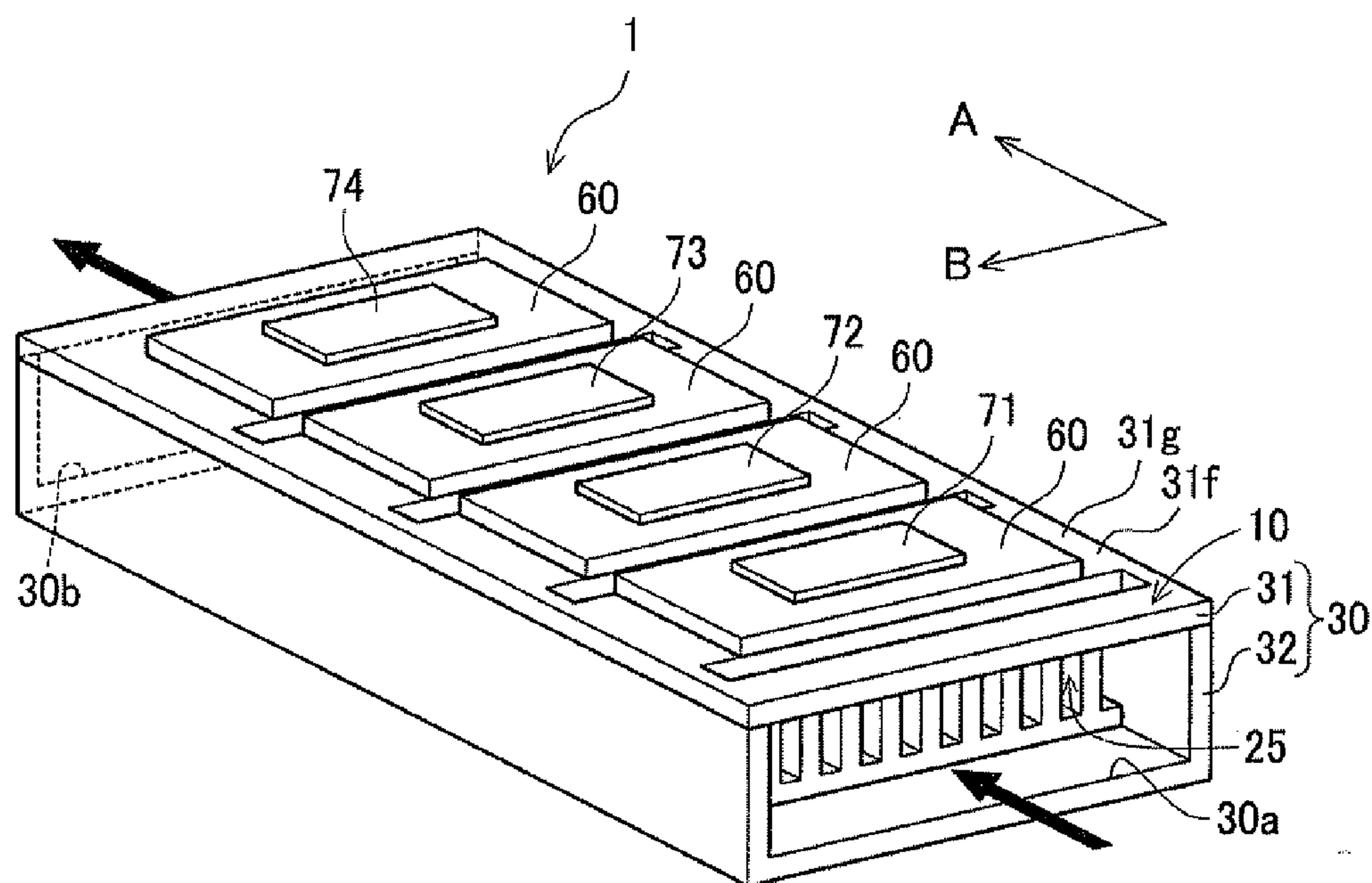


FIG.2

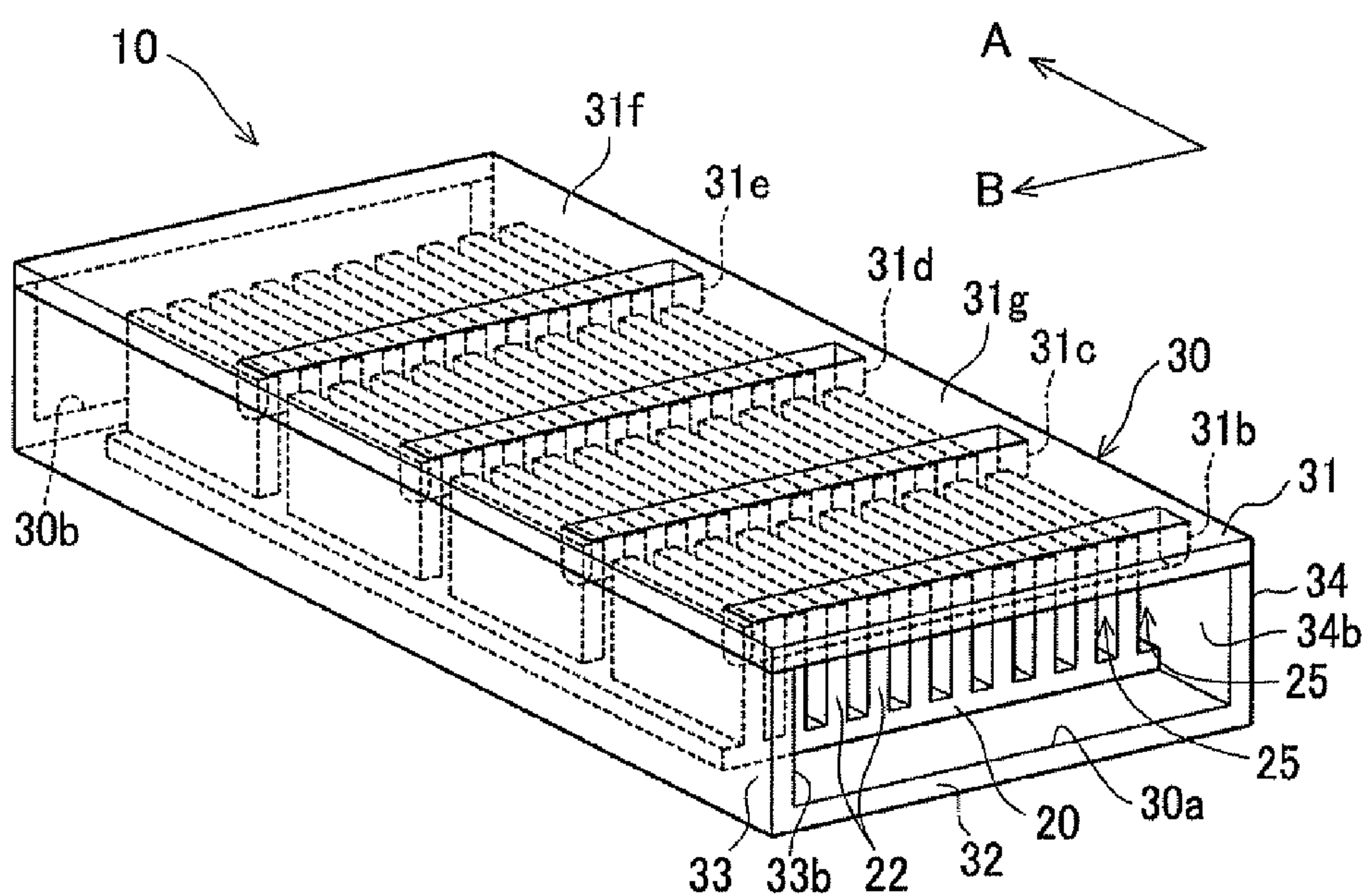


FIG.3

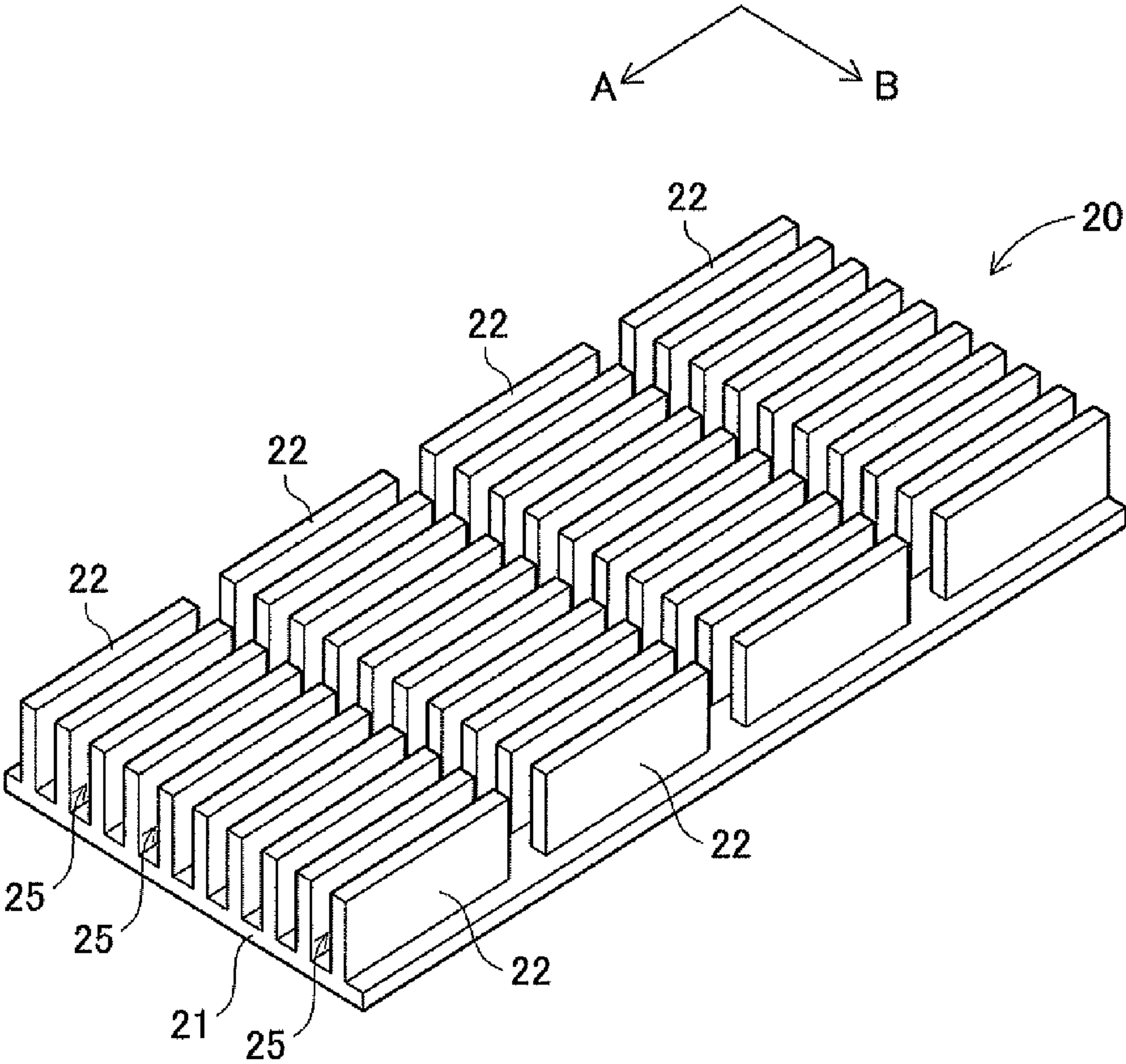


FIG.4

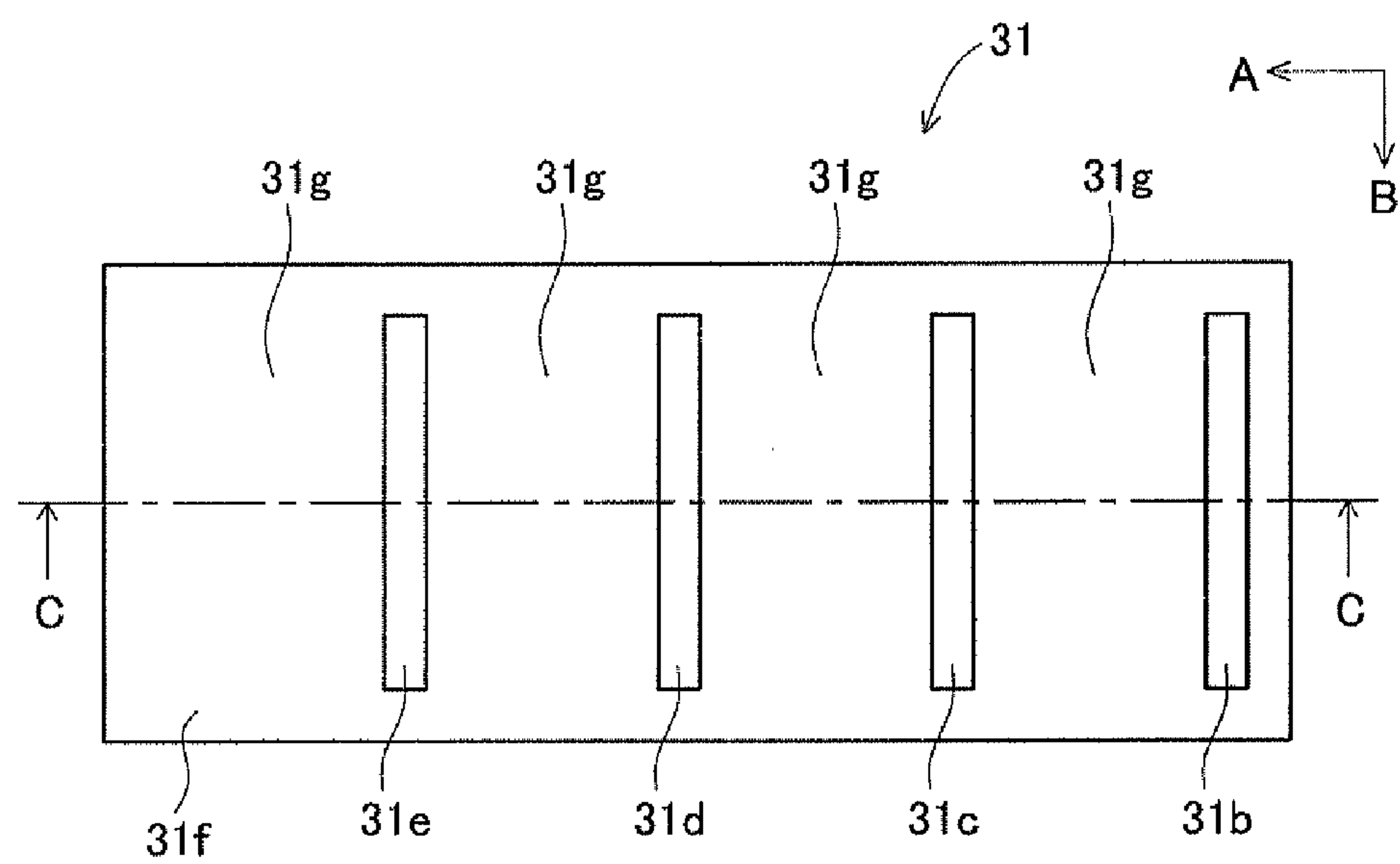
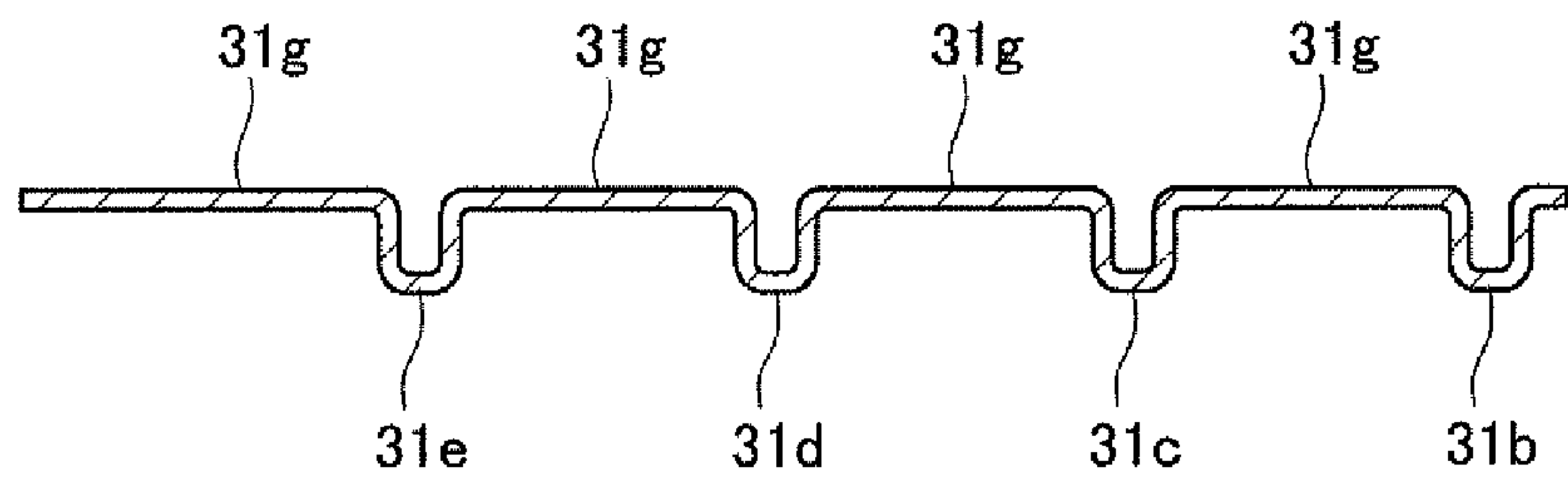


FIG.5





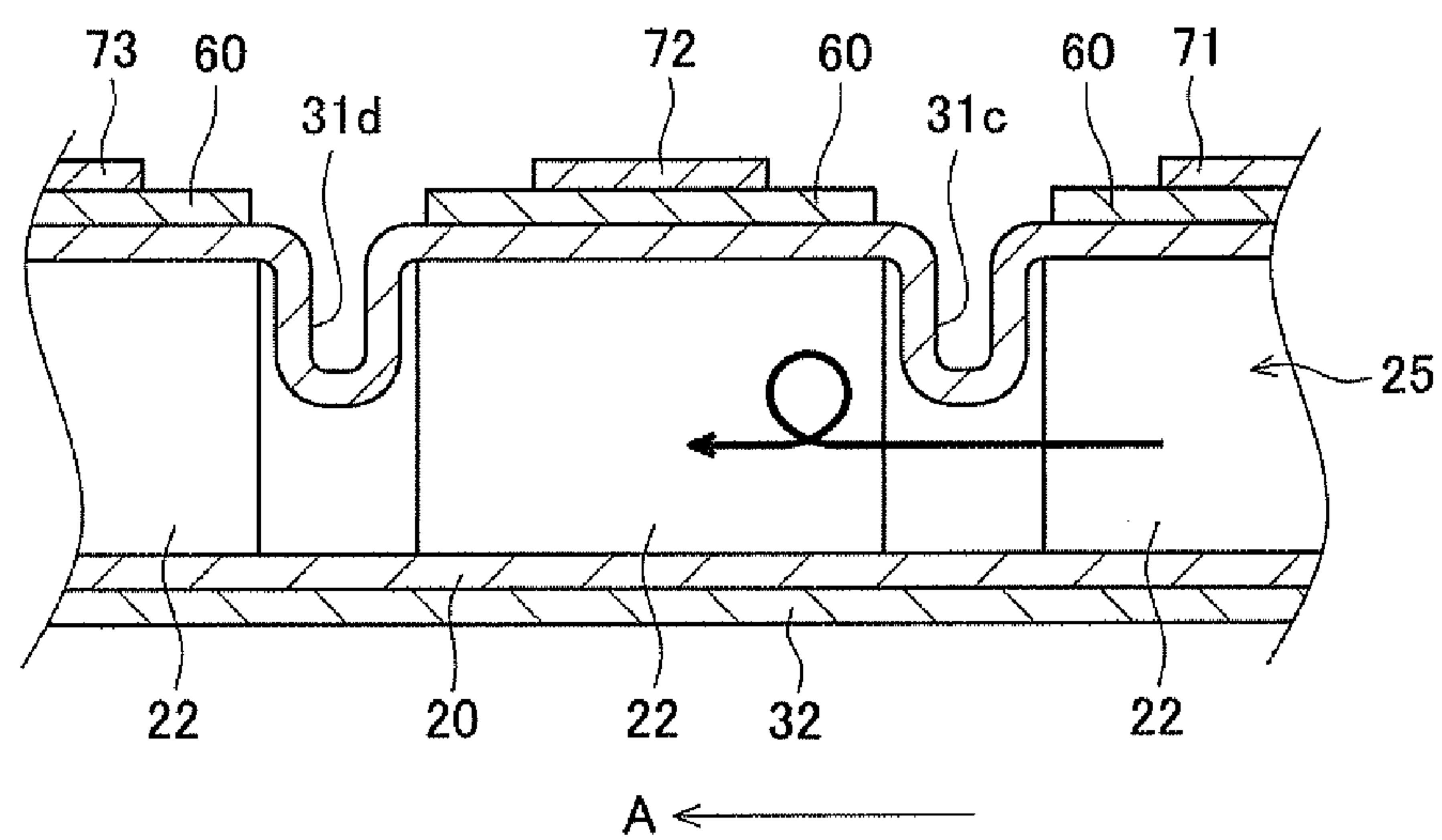


FIG.8

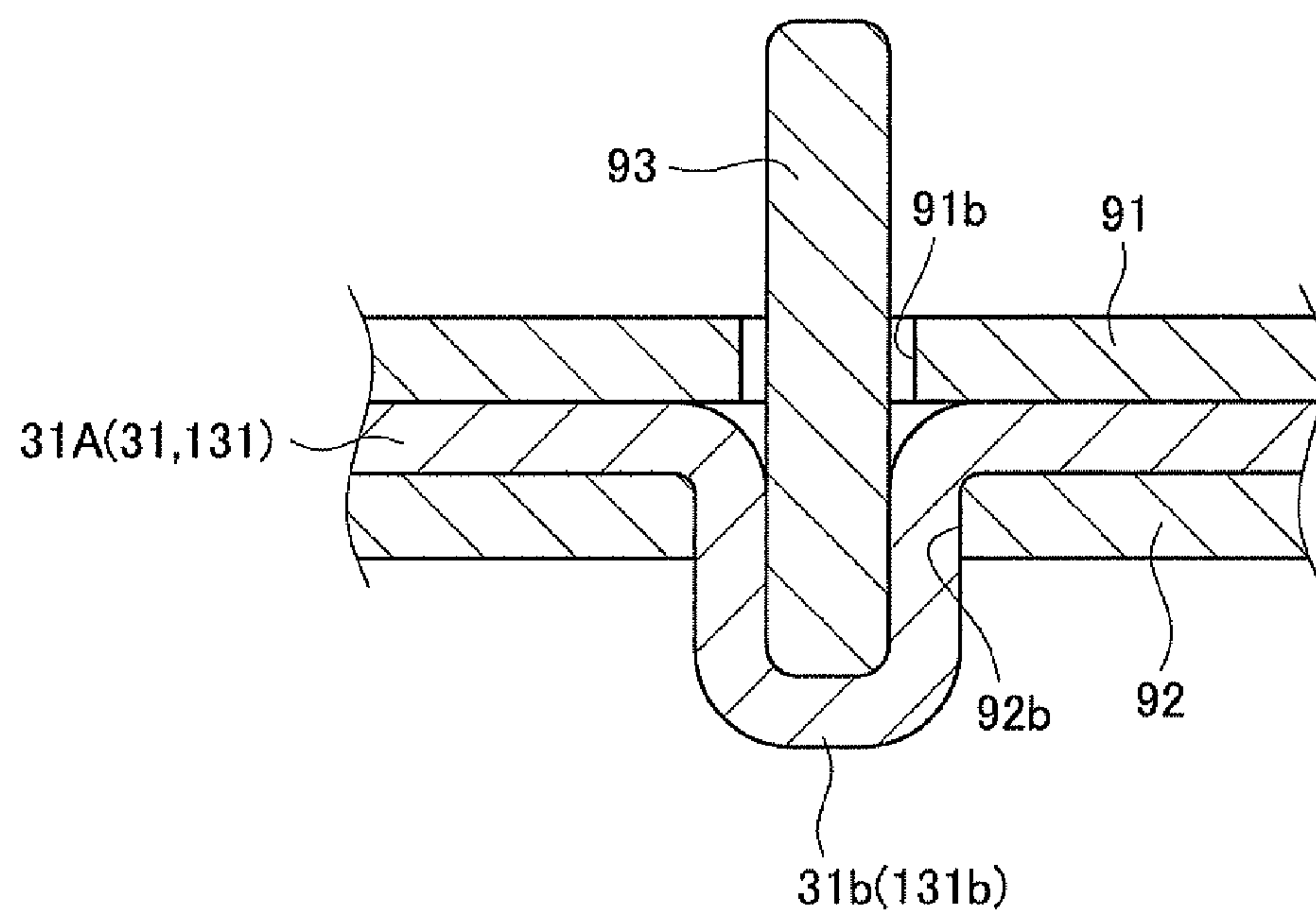


FIG.9

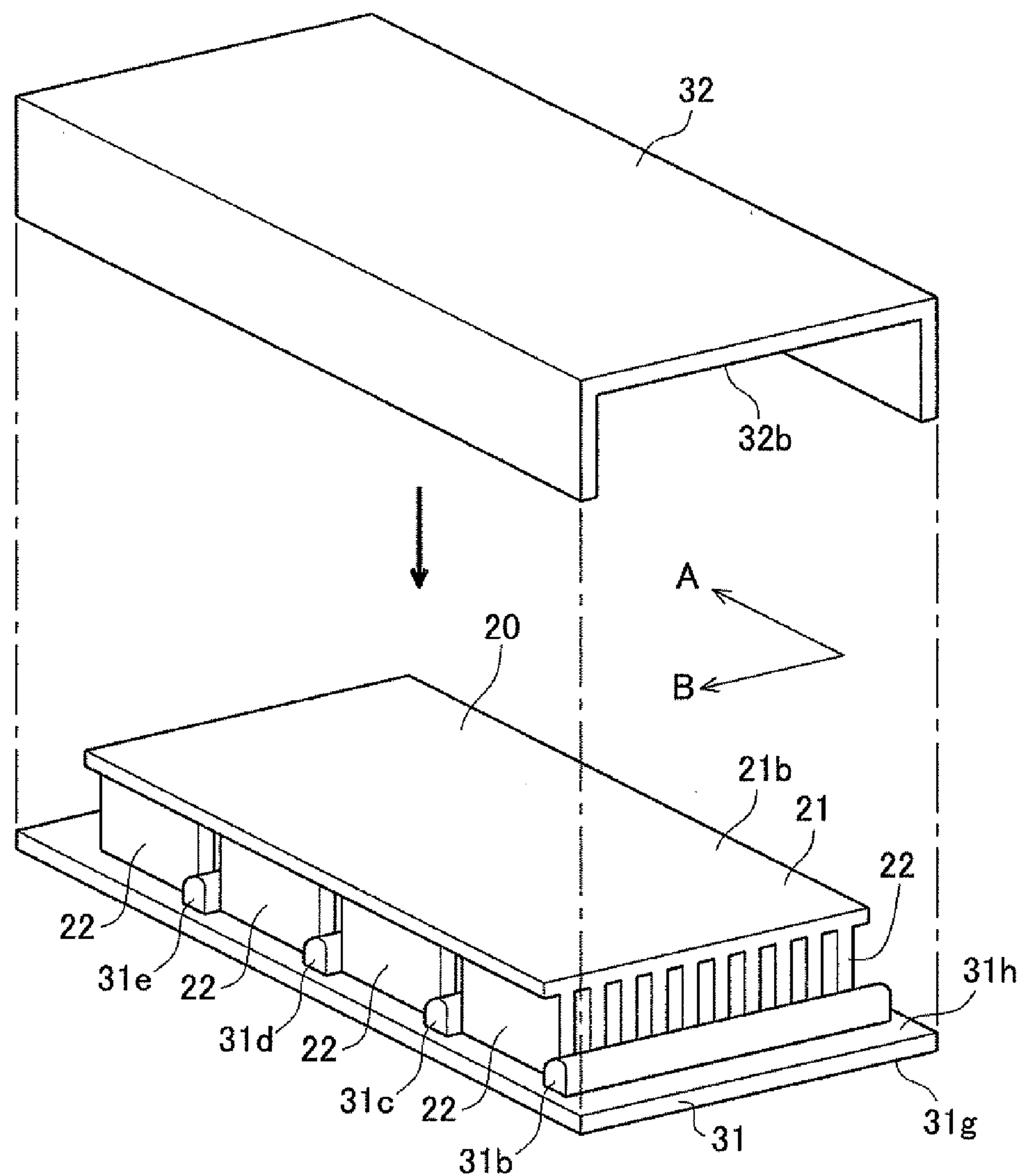


FIG.10

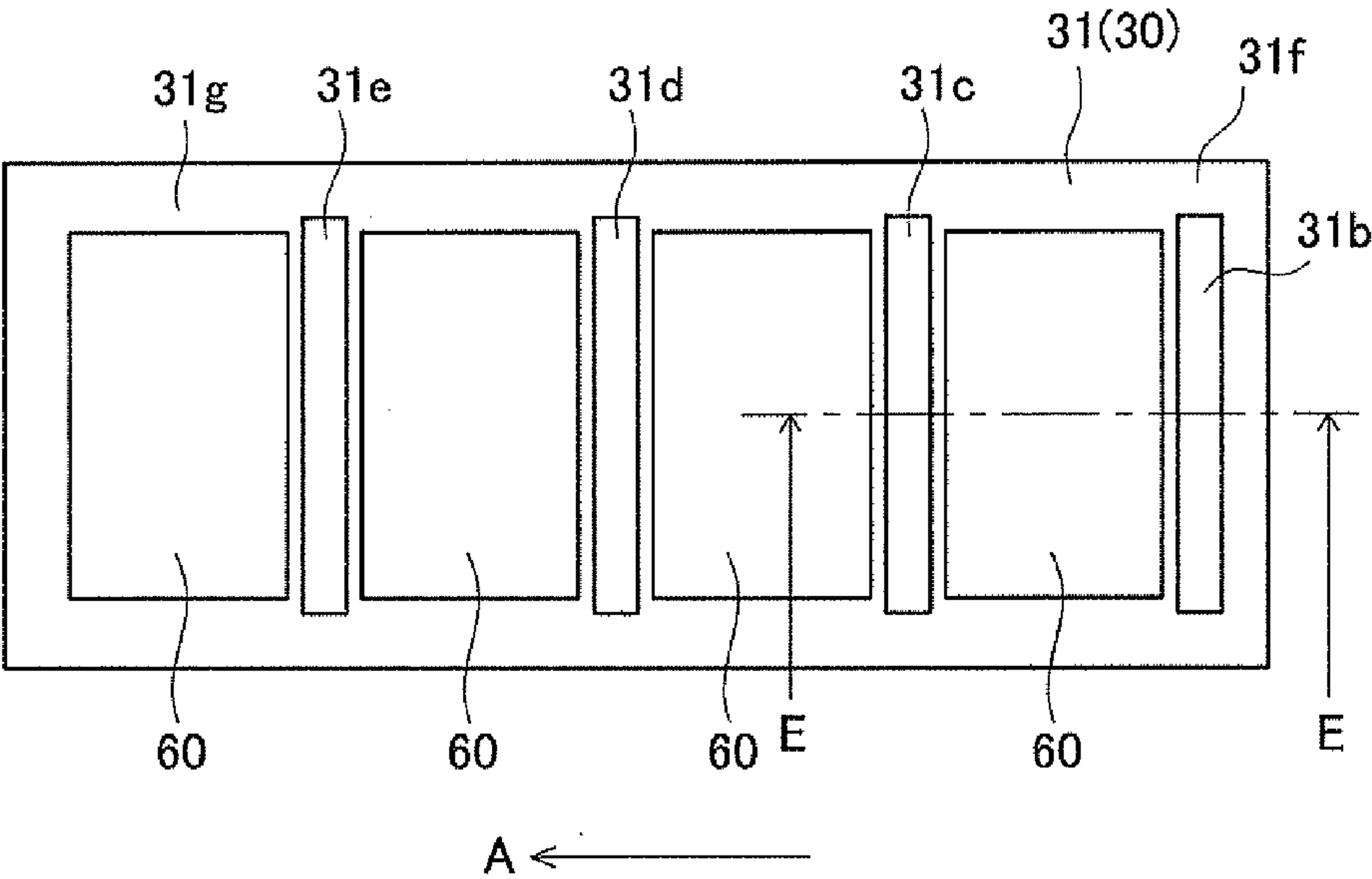


FIG.11

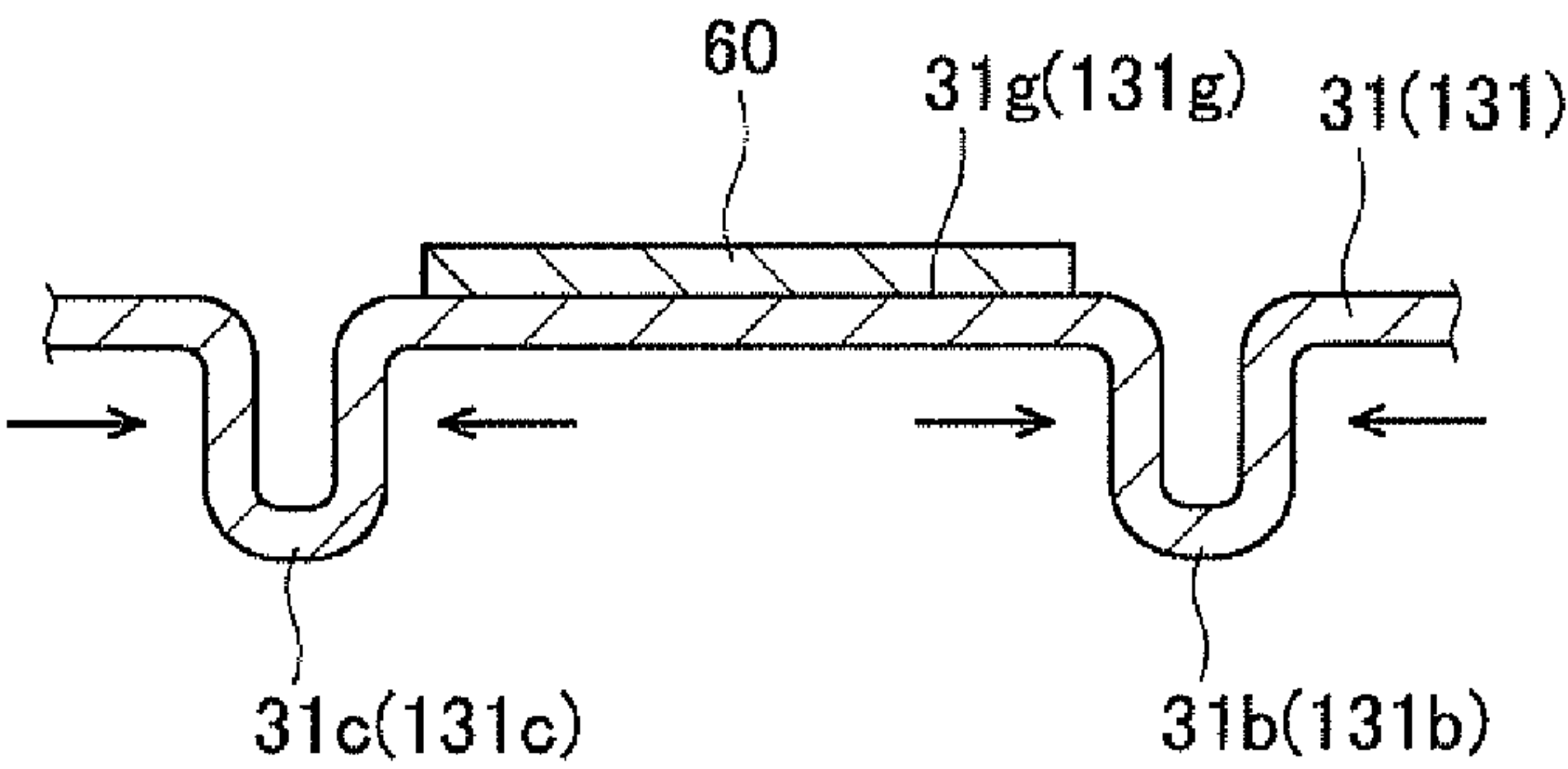




FIG.12

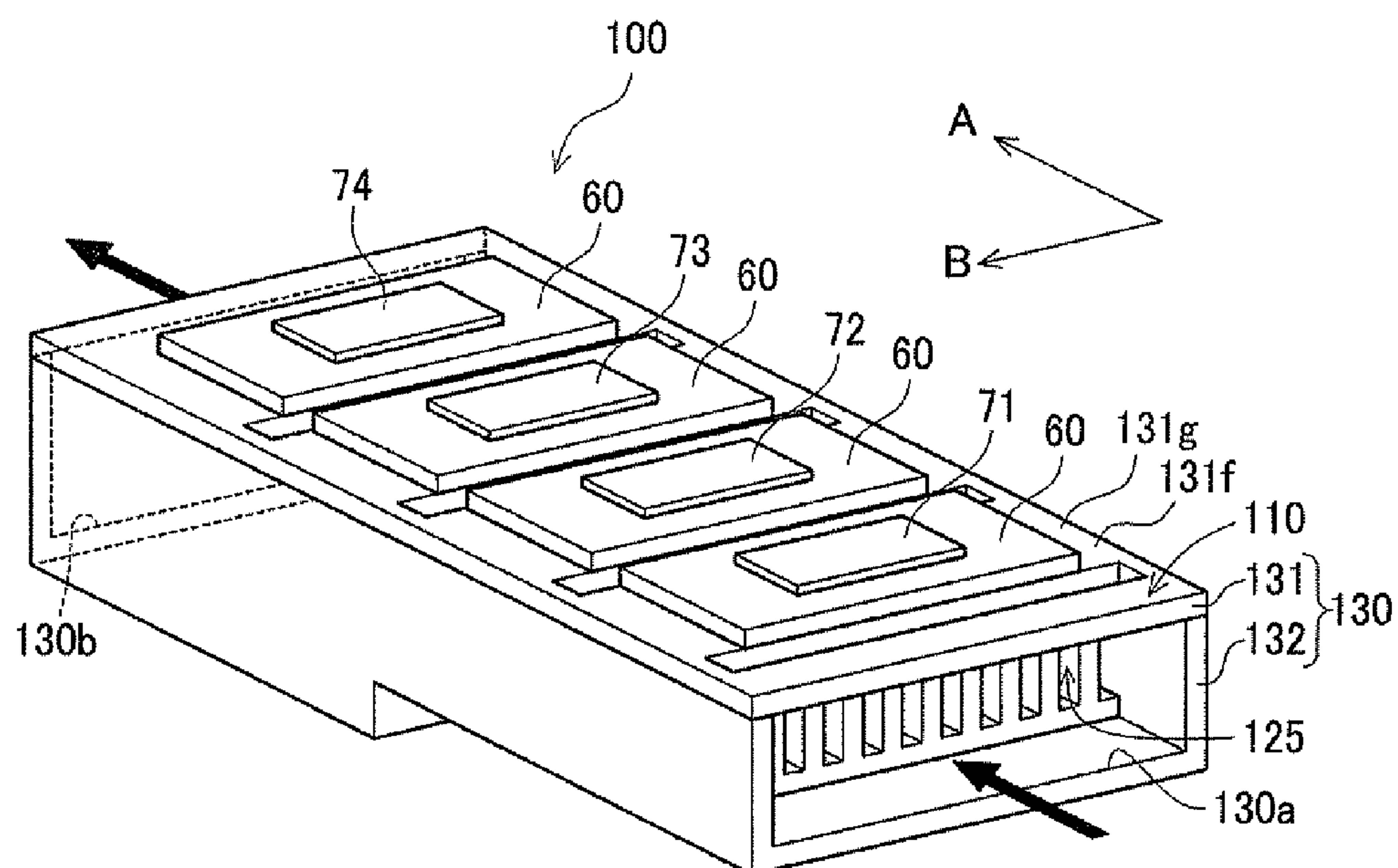


FIG.13

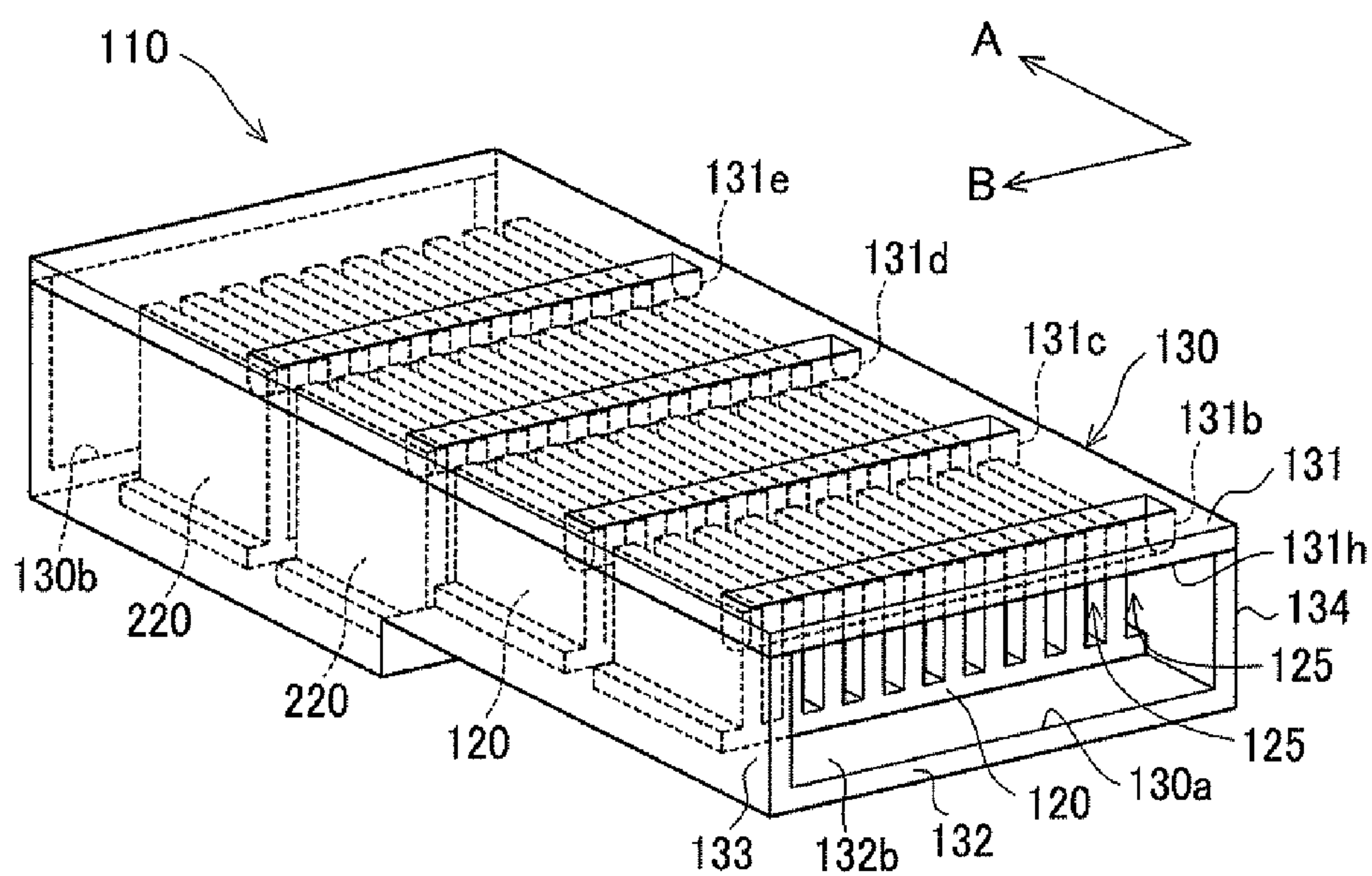


FIG.14

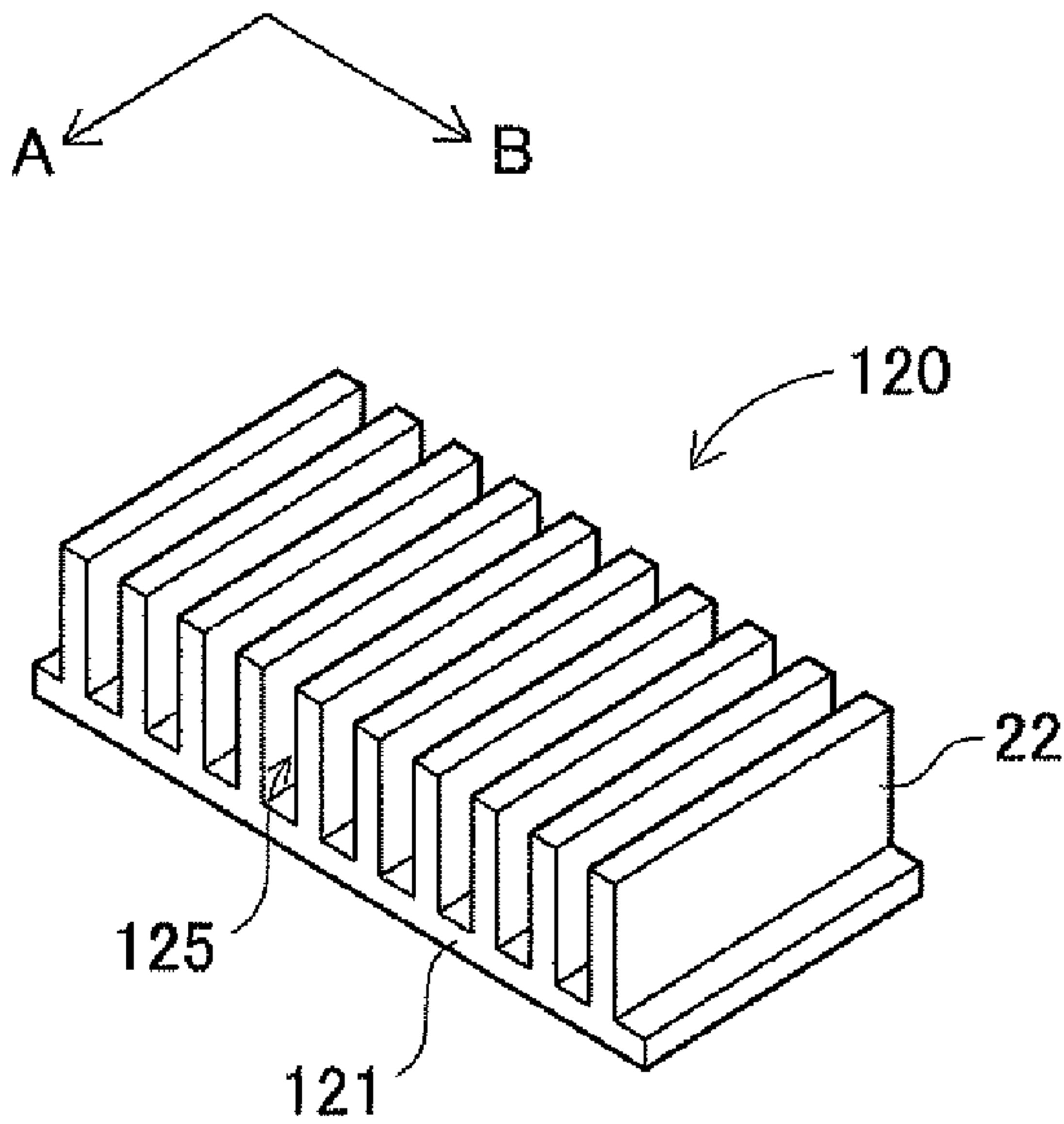


FIG.15

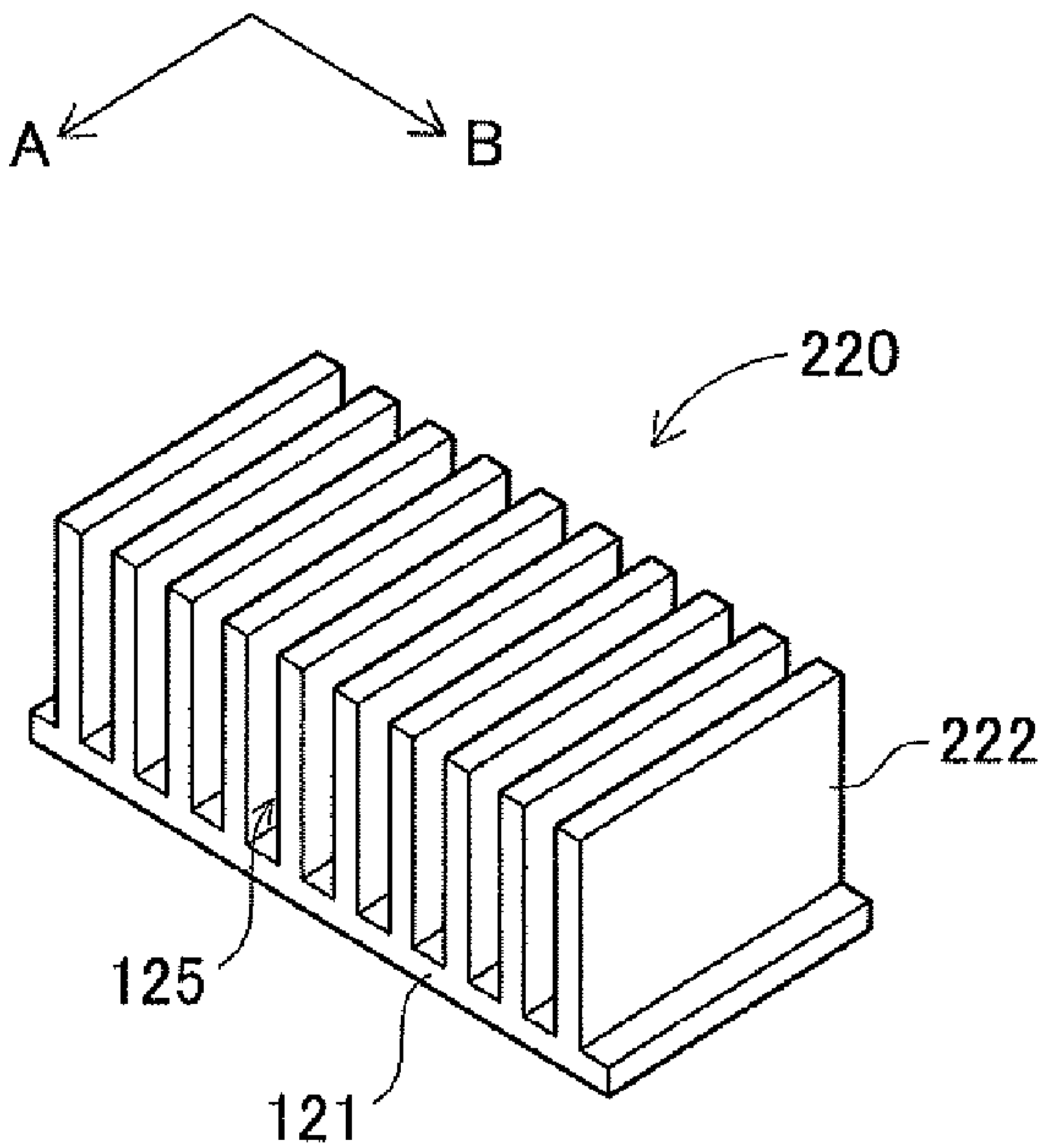


FIG.16

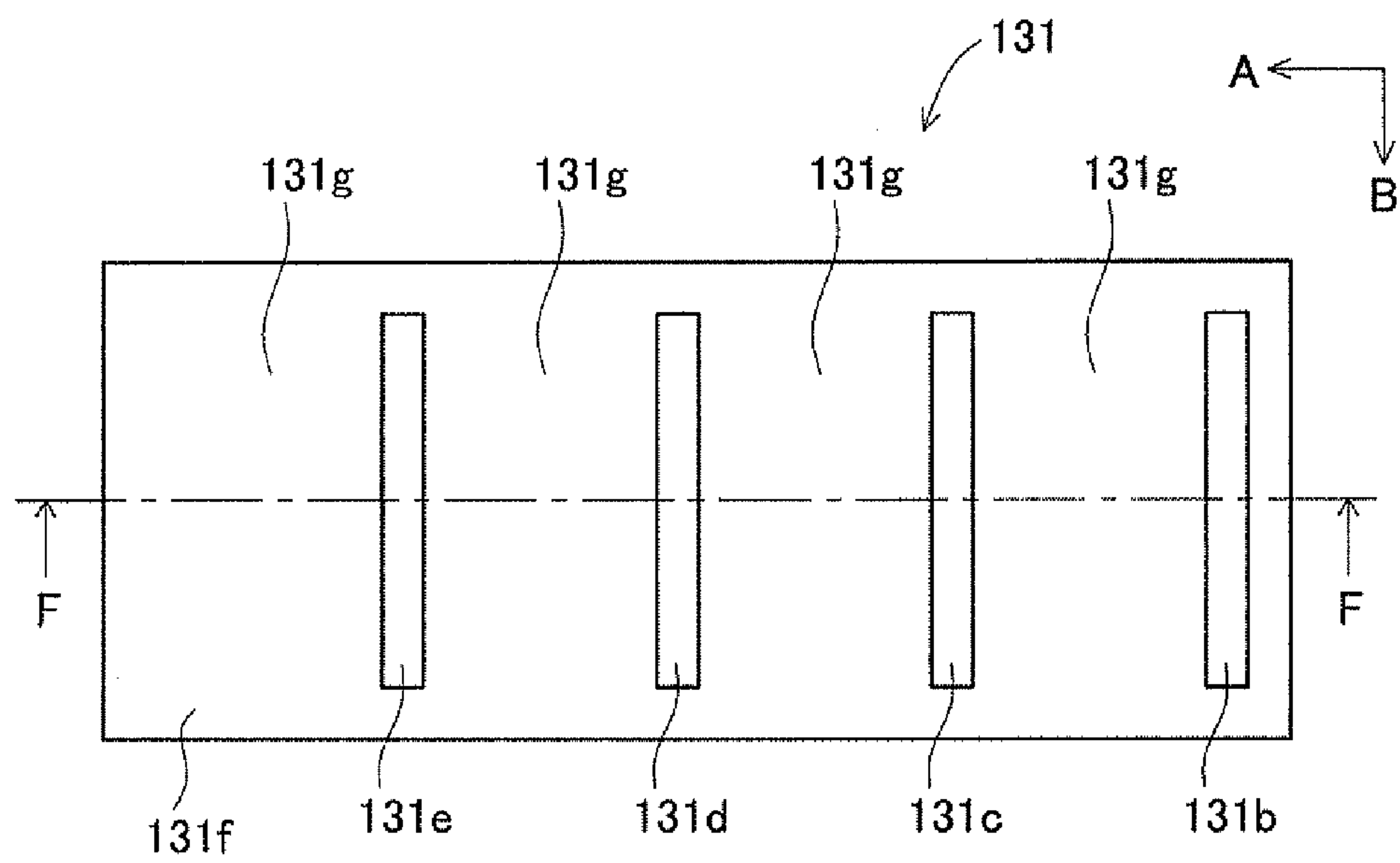


FIG.17

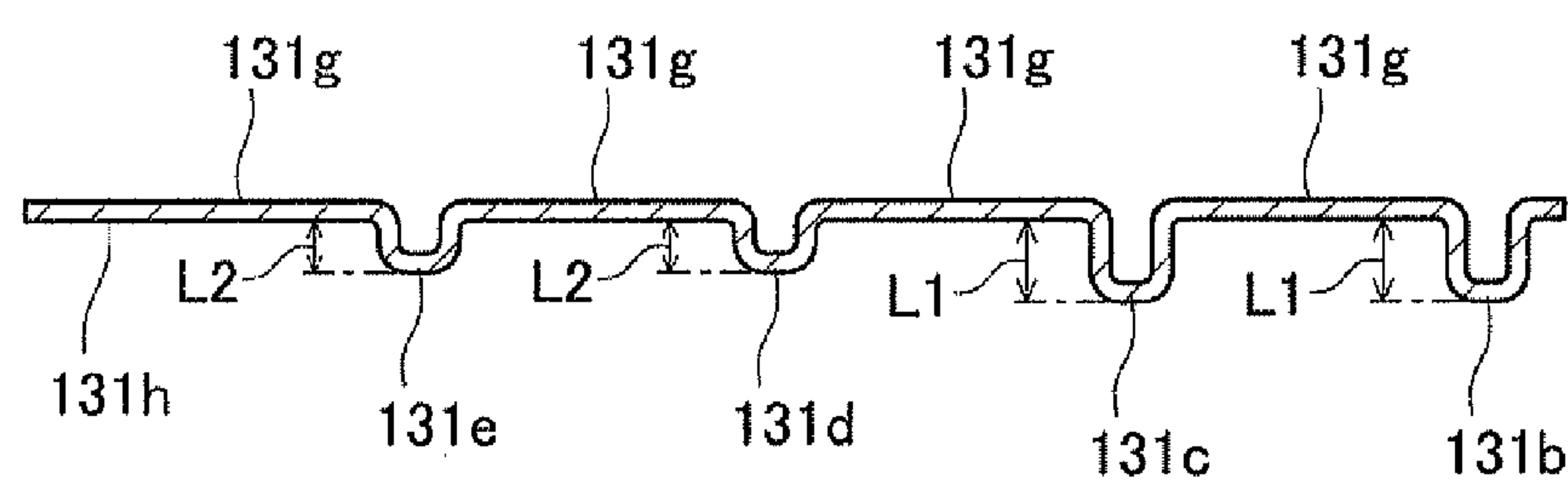


FIG.18

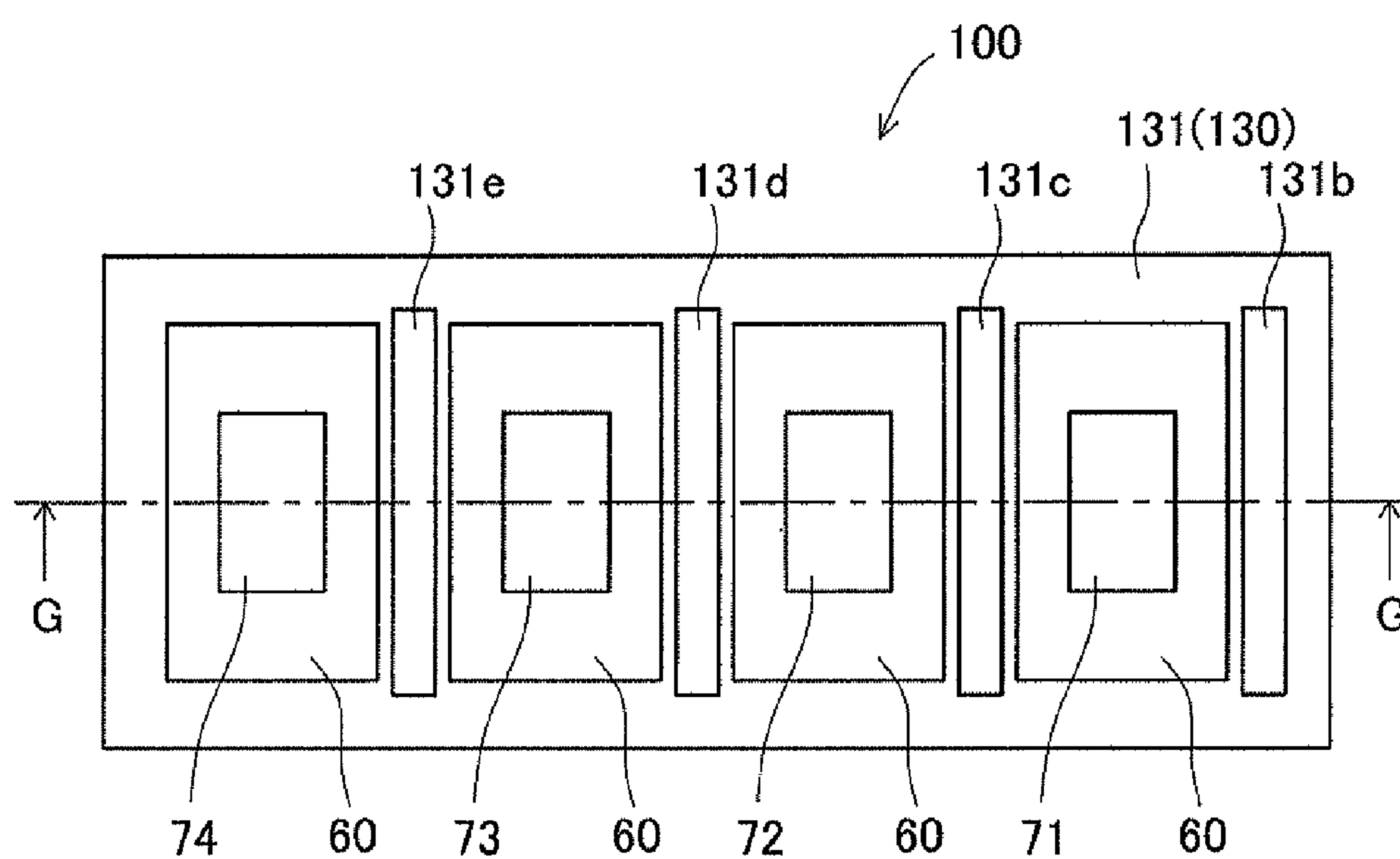
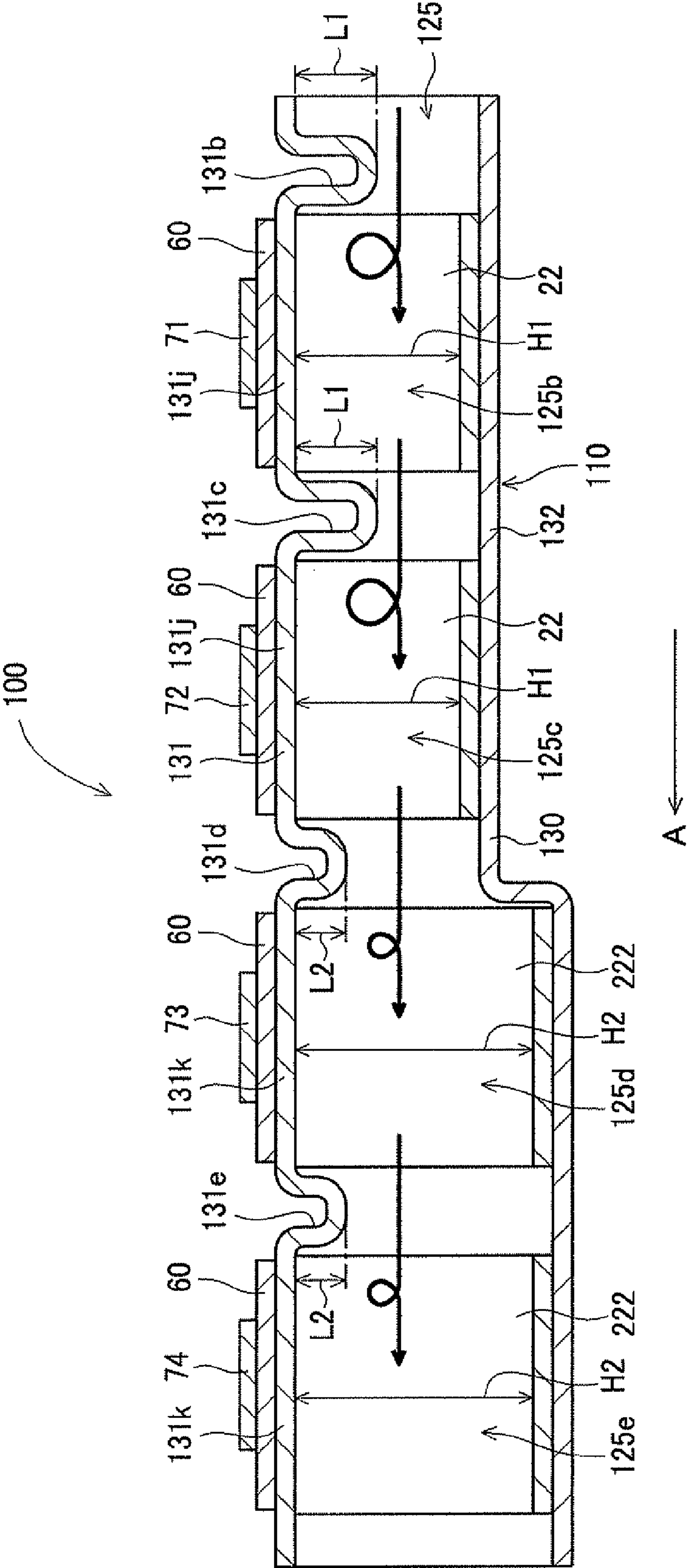


FIG.19





**HEAT EXCHANGER, SEMICONDUCTOR  
DEVICE, METHOD FOR MANUFACTURING  
THE HEAT EXCHANGER, AND METHOD  
FOR MANUFACTURING THE  
SEMICONDUCTOR DEVICE**

TECHNICAL FIELD

**[0001]** The present invention relates to a heat exchanger for cooling a heat generating body such as a semiconductor element by a coolant (cooling medium) flowing in the heat exchanger, a semiconductor device including the heat exchanger and a semiconductor element, and methods for manufacturing them.

BACKGROUND ART

**[0002]** Inverters having a power converting function are used as power sources of hybrid electric vehicles and others. Each of such inverters includes a plurality of semiconductor elements as a switching element. The semiconductor elements of the inverters generate heat in association with power conversion and the like and thus need to be cooled actively.

**[0003]** Here, as one of the heat exchangers for cooling the heat generating bodies such as semiconductor elements, there is known a heat exchanger configured such that a plurality of fins each linearly extending is arranged in parallel in a frame forming an outer casing, thereby forming coolant flow channels (e.g., see Patent Document 1).

RELATED ART DOCUMENTS

Patent Documents

**[0004]** Patent Document 1: JP 2007-335588A

DISCLOSURE OF THE INVENTION

Problems to be Solved by the Invention

**[0005]** Meanwhile, when the semiconductor elements are to be cooled by the heat exchanger, the heat exchanger should preferably be electrically insulated from the semiconductor elements. For this purpose, an insulator having an electrical insulation property may be interposed between the heat exchanger and the semiconductor elements. This insulator is welded (e.g., brazed) to a wall part of a frame (e.g., an aluminum frame) of the heat exchanger.

**[0006]** However, the insulator and the frame are likely to have different linear expansion coefficients. Accordingly, during welding (e.g., brazing) of the insulator and the wall part of the frame, when the insulator and the wall part of the frame are heated and then cooled, the insulator and the frame may warp (bend) due to a difference in shrinkage rate (linear expansion coefficient) between the insulator and the wall part of the frame. This defect may be caused not only when the insulator is welded to the wall part of the frame but also when an intervening member having a different linear expansion coefficient from the frame, i.e., a member interposed between the wall part of the frame and the heat generating body such as the semiconductor element or the like, is welded to the wall part of the frame.

**[0007]** The present invention has been made in view of the circumstances and has a purpose to provide a heat exchanger and a method for manufacturing the heat exchanger, capable of restraining warping (bending) of an intervening member and a frame which are different in linear expansion coefficient when the intervening member and the wall part of the frame

are welded, and a semiconductor device with the intervening member and the frame prevented from warping (bending) and a method for manufacturing the semiconductor device.

Means of Solving the Problems

**[0008]** To achieve the above purpose, one aspect of the invention provides a heat exchanger for cooling a heat generating body, the heat exchanger including a frame forming an outer casing and a fin member having a plurality of fins forming flow channels for a coolant, the fin member being placed in the frame, wherein the frame includes a first wall part to which an intervening member is welded so that the intervening member is interposed between the frame and the heat generating body, the intervening member having a different linear expansion coefficient from a linear expansion coefficient of the frame, and the first wall part of the frame includes an elastically deformable portion configured to be deformable in a direction along a mounting surface of an outer surface of the first wall part, on which mounting surface the intervening member is placed.

**[0009]** In the aforementioned heat exchanger, the first wall part of the frame (the wall part to which the intervening member welded so that the intervening member is interposed between the frame and the heat generating body) includes the elastically deformable portion configured to be elastically deformable in the direction along the mounting surface of the outer surface of the first wall part, on which the intervening member is mounted. Therefore, during welding of the intervening member and the first wall part of the frame that are different in linear expansion coefficient, when the intervening member and the first wall part are heated and then cooled, the elastically deformable portion is elastically deformed in the direction along the mounting surface according to a difference in shrinkage rate (linear expansion coefficient) between the first wall part and the intervening member. This can prevent the warping (bending) caused by the different in shrinkage rate (linear expansion coefficient) between the first wall part and the intervening member.

**[0010]** In the present application, “welding” represents a joining method achieved by heating and melting, such as brazing with brazing filler metal, soldering with solder, a joining method by melting a base material (a joining target member), and others.

**[0011]** The heat generating body includes all components that generate heat in use, such as a semiconductor element.

**[0012]** In the aforementioned heat exchanger, preferably, the elastically deformable portion includes a protrusion protruding into the frame, the protrusion having a U-shaped groove configuration having a U-shaped cross section that opens outward of the frame and linearly extending along the mounting surface in a direction intersecting with a coolant flowing direction.

**[0013]** In the aforementioned heat exchanger, the protrusion protruding into the frame constitutes the elastically deformable portion. This protrusion has the U-groove configuration having the U-cross section that opens outward of the frame and linearly extending along the mounting surface of the first wall part. In other words, the protrusion is formed in a U-groove shape having the U-cross section that opens outward of the frame and linearly extending the direction (corresponding to a direction perpendicular to the cross section) along the mounting surface of the first wall part. Such configured protrusion is elastically deformable in the direction along the mounting surface of the first wall part by



widening (expanding) and narrowing (contracting) the opening of the U-groove. Therefore, this protrusion is elastically deformed in the direction along the mounting surface of the first wall part, thereby preventing the warping tending to occur due to the difference in shrinkage rate (linear expansion coefficient) between the first wall part and the intervening member.

**[0014]** Meanwhile, according to an investigation of the heat exchanger with the fin member including the fins forming coolant flow channels in the frame constituting the outer casing, a flow velocity distribution of coolant flowing in clearances between the fins has a tendency of the coolant flow velocity being slower as it comes closer to the fins. This is because the coolant is attracted by the fins due to the viscosity of coolant. As a result, in a region near the fins, the flow velocity of coolant is slower than in other regions or the coolant hardly moves. This region is also referred hereinafter to as a boundary layer. When this boundary layer is generated, the fins that undergo heat exchange with only the coolant staying in the boundary layer generated mostly around the fins. Thus, heat exchange with the coolant flowing in other regions than the boundary layer is hardly conducted. Consequently, heat exchange with the coolant flowing inside the heat exchanger is not effectively performed and a high cooling effect could not be achieved.

**[0015]** On the other hand, in the aforementioned heat exchanger, the protrusion protruding into the frame linearly extends in the direction intersecting with the coolant flowing direction. This configuration can cause the coolant flowing in the flow channel to collide with the protrusion and thereby agitate the coolant. Accordingly, the flow of coolant is made turbulent, thus effectively preventing the occurrence of the boundary layer. It is therefore possible to effectively utilize the coolant flowing in the heat exchanger to obtain a high cooling effect.

**[0016]** In the aforementioned heat exchanger, preferably, the protrusion is integrally formed in the wall part by drawing.

**[0017]** In the aforementioned heat exchanger, the protrusion is integrally formed in the first wall part by drawing. Since the protrusion is integrally formed in the first wall part by drawing, the protrusion can be formed easily and at low costs. Thus, the aforementioned heat exchanger is low in cost.

**[0018]** In one of the aforementioned heat exchangers, preferably, the protrusion is provided to extend to inner wall surfaces of both side walls of the frame in the direction intersecting with the coolant flowing direction.

**[0019]** The protrusion protruding into the frame is provided to the inner wall surfaces of both side walls of the frame in the direction intersecting with the coolant flowing direction. This configuration can reliably make the coolant flowing in all the flow channels formed between the fins collide with the protrusions and agitate the coolant. Accordingly, the coolant flowing in all the flow channels can be surely made turbulent, thereby effectively prevent the occurrence of the boundary layer. This configuration can more effectively utilize the coolant flowing in the heat exchanger and provide a higher cooling effect.

**[0020]** In one of the aforementioned heat exchangers, preferably, the protrusion linearly extends in a direction perpendicular to the coolant flowing direction.

**[0021]** Since the protrusion is configured to linearly extend in the direction intersecting with the coolant flowing direction, maximum flowing resistance can be created in the direction perpendicular to the coolant flowing direction. This con-

figuration can generate the turbulent flows most effectively, thereby more reliably enhancing the cooling effect.

**[0022]** In one of the aforementioned heat exchangers, preferably, the protrusion includes protrusions arranged side by side at predetermined intervals in the coolant flowing direction, and the fins are placed between the protrusions located adjacently in the coolant flowing direction.

**[0023]** The fins are arranged between the protrusions adjacently located in the coolant flowing direction and therefore the coolant flowing in the flow channels formed between the fins can be made to appropriately collide with the protrusions and thereby agitated. This configuration can prevent the occurrence of the boundary layer and achieve a high cooling effect.

**[0024]** In the aforementioned heat exchanger, preferably, the heat exchanger is configured to cool a plurality of heat generating bodies arranged side by side in the coolant flowing direction, the first wall part includes a low-heat generation portion on which a heat generating body having a relatively small heat generation amount among the heat generating bodies is placed through the intervening member, and, of the protrusions, a protrusion located upstream in the flow channel from and adjacent to the low-heat generation portion has a shorter protruding height protruding into the frame than a protruding height of other protrusion.

**[0025]** As the protruding height (length) of the protrusions is higher, the flowing resistance of the coolant flowing in the flow channels can be more increased. Thus, the ability of agitating the coolant flowing in the flow channels can be enhanced, and the effect of preventing the occurrence of the boundary layer can be enhanced. However, if the protruding height of the protrusions is too high, pressure loss in the flow channels becomes too large, leading to a great decrease in flow amount of the coolant. This rather decreases the cooling effect.

**[0026]** Meanwhile, when the heat generating bodies (semiconductor elements and others) different in heat generation amount are to be cooled, it is unnecessary to cool all the heat generating bodies uniformly but each heat generating body should be preferably cooled according to respective heat generation amounts. Accordingly, among the heat generating bodies, some heat generating bodies having a relatively small heat amount may be less cooled than other heat generating bodies (the heat generating bodies having a relatively large heat amount).

**[0027]** Therefore, in the aforementioned heat exchanger, among the protrusions, the protrusion located upstream in the flow channel from and adjacent to the low-heat generation portion (the portion of the first wall part, on which portion the heat generating body having a relatively small heat amount among the heat generating bodies is placed with the intervening member interposed) is designed with a shorter protruding height of the frame internally protruding than a protruding height of other protrusion. Since some of the protrusions have such low protruding height, pressure loss in the flow channels can be restrained.

**[0028]** In contrast, the protrusion with low protruding height is inferior in coolant agitating ability as compared with other protrusions. Thus, the flow channels located adjacent to and downstream of the protrusion with low protruding height is poor at preventing the occurrence of the boundary layer. In the aforementioned heat exchanger, in contrast, the protrusion located upstream in the flow channel from and adjacent to the low-heat generation portion is designed with lower



protruding height. Specifically, the first wall part includes a low-heat generation portion located adjacent to and downstream of the protrusion with low protruding height (on a downstream side in the flow channels). Accordingly, only the low-heat generation portion has a decreased cooling ability. As above, the cooling ability can be decreased in only the low-heat generation portion and thus each heat generating body can be appropriately cooled according to respective amounts of heat generation.

**[0029]** In the aforementioned heat exchanger, preferably, a flow channel section of the coolant flow channel, the flow channel section including the low-heat generation portion, is deeper in flow channel depth than other flow channel sections are.

**[0030]** In the aforementioned heat exchanger, the coolant flow channel is configured such that the flow channel section defined to include the low-heat generation portion is deeper in flow channel depth (flow channel depth) than other flow channels. The flowing resistance of the coolant flowing in the flow channel can be reduced by the increased depth of the flow channel section including the low-heat generation portion. Accordingly, pressure loss in the flow channel can be further reduced.

**[0031]** In the flow channel section with deeper flow channel depth, the flow velocity becomes lower than in the other flow channel section, resulting in a decrease in cooling ability. In contrast, in the aforementioned heat exchanger, only the flow channel section including the low-heat generation portion is deeper than others. Since only the low-heat generation portion is decreased in cooling ability, each heat generating body can be appropriately cooled according to respective heat generation amounts.

**[0032]** Another aspect of the invention provides a semiconductor device including: one of the aforementioned heat exchangers; a semiconductor element that is the heat generating body; and the intervening member welded to the first wall part of the frame and located between the semiconductor element and the first wall part.

**[0033]** The aforementioned semiconductor device include the heat exchanger mentioned above. Accordingly, the semiconductor device can prevent warping (bending) of the intervening member and the frame.

**[0034]** Another aspect of the invention provides a method for manufacturing a heat exchanger for cooling a heat generating body, the heat exchanger including a frame forming an outer casing and a fin member having a plurality of fins forming flow channels for coolant, the fin member being placed in the frame, the method comprising: a forming step of forming the frame including a first wall part to which an intervening member to be interposed between the frame and the heat generating body is welded; and an assembling step of assembling the heat exchanger by placing the fin member in the frame formed in the forming step, the forming step including drawing to form the protrusion protruding into the frame integral with the first wall part so that the protrusion has a U-shaped groove configuration having a U-shaped cross section that opens outward of the frame and linearly extending along a mounting surface on which the intervening member is to be placed, of the outer surface of the first wall part.

**[0035]** In the above method for manufacturing the heat exchanger, the forming step of forming the frame includes integrally forming the protrusion protruding into the frame in the first wall part (the wall part to which the intervening member is welded) by drawing. As above, the protrusion is

integrally formed in the first wall part by drawing and the protrusion can be formed easily and at low costs.

**[0036]** In addition, this protrusion is formed in the U-like groove shape having a U-cross sectional that opens outward of the frame and linearly extending along the mounting surface of the outer surface of the first wall part, on which mounting surface the intervening member is placed. Such configured protrusion has the opening of the U-like groove expands and contracts so that the protrusion be elastically deformed in the direction along the mounting surface of the first wall part. Accordingly, when the intervening member is welded to the first wall part, this protrusion is elastically deformed according to a difference in shrinkage rate (linear expansion coefficient) between the first wall part and the intervening member. It is therefore possible to restrain the warping caused by the difference in shrinkage rate (linear expansion coefficient) between the first wall part and the intervening member.

**[0037]** Another aspect of the invention provides a method for manufacturing a semiconductor device comprising: the heat exchanger manufactured by the aforementioned heat exchanger manufacturing method; a semiconductor element that is the heat generating body; and an intervening member to be interposed between the first wall part of the frame and the semiconductor element, the intervening member having a different linear expansion coefficient from a linear expansion coefficient of the frame, the method comprising: a welding step of placing the intervening member on the mounting surface of the first wall part of the frame and then welding the intervening member to the first wall part.

**[0038]** In the above method for manufacturing the semiconductor device, the welding step includes welding the frame and the intervening member different in linear expansion coefficient to the first wall part of the frame. In a conventional manufacturing method, during welding of an intervening member (an insulator) and a wall part of a frame, when the insulator and the frame wall part are heated and then cooled, the intervening member and the frame may warp (bend) due to a difference in shrinkage rate (linear expansion coefficient) between the intervening member and the frame wall part.

**[0039]** In contrast, the above manufacturing method uses the frame integrally including the first wall part and the protrusion having a U-shaped cross section that opens outward of the frame and a U-groove shape linearly extending in the direction along the mounting surface of the outer surface of the first wall part, on which the intervening member is mounted. Therefore, as mentioned above, when the intervening member is welded to the first wall part, the protrusion is elastically deformed in the direction along the mounting surface of the first wall part according to the difference in shrinkage (linear expansion coefficient) between the first wall part and the intervening member. This can prevent the warping caused by the difference in shrinkage (linear expansion coefficient) between the first wall part and the intervening member.

**[0040]** It is to be noted that a method for welding the intervening member with the first wall part may include brazing using a brazing filler metal, for example.

#### BRIEF DESCRIPTION OF THE DRAWINGS

**[0041]** FIG. 1 is a perspective view of a semiconductor device in a first embodiment;

**[0042]** FIG. 2 is a perspective view of a heat exchanger in the first embodiment;



[0043] FIG. 3 is a perspective view of a fin member of the heat exchanger in the first embodiment;

[0044] FIG. 4 is a plan view of a first frame part of the heat exchanger in the embodiment;

[0045] FIG. 5 is a cross sectional view of the first frame part, taken along a line C-C in FIG. 4;

[0046] FIG. 6 is a plan view of the semiconductor device in the first embodiment;

[0047] FIG. 7 is a cross sectional view of the semiconductor device in the first embodiment, taken along a line D-D in FIG. 6;

[0048] FIG. 8 is an explanatory view showing a forming process (drawing) in the first and second embodiments;

[0049] FIG. 9 is an explanatory view showing an assembling process in the first embodiment;

[0050] FIG. 10 is an explanatory view showing a welding process in the first embodiment;

[0051] FIG. 11 is an explanatory view showing actions of a protrusion in the welding process in the first embodiment;

[0052] FIG. 12 is a perspective view of a semiconductor device in the second embodiment;

[0053] FIG. 13 is a perspective view of a heat exchanger in the second embodiment;

[0054] FIG. 14 is a perspective view of a fin member of the heat exchanger in the second embodiment;

[0055] FIG. 15 is a perspective view of another fin member of the heat exchanger in the second embodiment;

[0056] FIG. 16 is a plan view of a first frame part of the heat exchanger in the second embodiment;

[0057] FIG. 17 is a cross sectional view of the first frame part, taken along a line F-F in FIG. 16;

[0058] FIG. 18 is a plan view of the semiconductor device in the second embodiment; and

[0059] FIG. 19 is a cross sectional view of the semiconductor device in the second embodiment, taken along a line G-G in FIG. 18.

## MODE FOR CARRYING OUT THE INVENTION

### Example 1

[0060] A detailed description in Example 1 of the present invention will now be given referring to the accompanying drawings.

[0061] A semiconductor device in Example 1 includes semiconductor elements 71-74 (corresponding to heat generating bodies) and a heat exchanger 10 to cool the semiconductor elements as shown in FIG. 1.

[0062] A heat exchanger 10 includes, as shown in FIG. 2, a frame 30 forming an outer casing and a fin member 20 accommodated in the frame 30. These frame 30 and fin member 20 are joined together by brazing.

[0063] In FIGS. 1 and 2, a direction A indicates a flowing direction of a coolant (e.g., water) flowing in the heat exchanger 10 and a direction B indicates a direction perpendicular to the direction A.

[0064] The fin member 20 is made of aluminum and includes, as shown in FIG. 3, a rectangular plate-like base 21 and a plurality of fins 22 (forty fins in Example 1) each protruding from one surface of the base 21. Each fin 22 has a rectangular plate-like shape. In a long-side direction (corresponding to the direction A) of the base 21, four fins 22 are arranged in line at fixed intervals. Further, in a short-side direction (corresponding to the direction B) of the base 21, ten fins 22 are arranged in row at fixed intervals. Between each

clearance of the fins 22, flow channels 25 for guiding the coolant in the direction A is formed with a predetermined width. This fin member 20 can be integrally made of aluminum by extrusion, for example.

[0065] The frame 30 includes a first frame part 31 made of aluminum in a rectangular plate-like shape and a second frame part 32 made of aluminum in a rectangular U-shaped configuration (see FIG. 2). The first frame part 31 and the second frame part 32 are joined to each other by brazing. Thus, the frame 30 has a rectangular tubular shape. This frame 30 includes an inlet 30a at one end in a longitudinal direction (corresponding to the direction A) to allow inflow of the coolant into the frame 30 and an outlet 30b at the other end in the longitudinal direction (corresponding to the direction A) to discharge the coolant.

[0066] The first frame part 31 has an outer surface 31f including flat mounting surfaces 31g in four locations on which insulating plates 60 (corresponding to intervening members) are placed respectively between the frame 30 and the semiconductor elements 71 to 74 (see FIGS. 1, 4, and 5). Each insulating plate 60 is made of an electrically insulating member (e.g., ceramics such as alumina) formed in a rectangular flat plate-like shape. Four insulating plates 60 placed on respective mounting surfaces 31g are arranged in one line at equal intervals in the longitudinal direction (corresponding to the direction A) of the first frame part 31 (see FIGS. 1 and 6). Those insulating plates 60 are joined to the mounting surfaces 31g of the first frame part 31 by brazing. In Example 1, the first frame part 31 corresponds to the first wall part.

[0067] Further, the first frame part 31 includes a plurality of protrusions (four protrusions in Example 1) 31b, 31c, 31d, and 31e each protruding into the frame 30 (see FIGS. 2 and 5). Those protrusions 31b to 31e have the same shape. Specifically, as shown in FIGS. 4 and 5, each of the protrusions 31b to 31e has a U-shaped groove configuration having a U-shaped cross section that opens outward (upward in FIG. 5) of the frame 30 and linearly extending in the direction B along the mounting surfaces 31g, i.e., in the direction perpendicular to the coolant flowing direction and in a vertical direction in FIG. 4. Those protrusions 31b to 31e are integrally formed in the first frame part 31 by drawing.

[0068] In the first frame part 31, the mounting surfaces 31g are respectively present between the protrusions 31b and 31c, between the protrusions 31c and 31d, between the protrusions 31d and 31e, and in a forward adjacent position to the protrusion 31e in the direction A.

[0069] Meanwhile, the insulating plates 60 and the frame 30 (the first frame part 31) have different linear expansion coefficients. To be concrete, for example, each insulating plate 60 made of alumina has a linear expansion coefficient of about  $7 \times 10^{-6}/^{\circ}\text{C}$ . On the other hand, the linear expansion coefficient of the frame 30 (the first frame part 31) made of aluminum is about  $23 \times 10^{-6}/^{\circ}\text{C}$ . In this case, the linear expansion coefficient of the frame 30 (the first frame part 31) is more than three times as high as the linear expansion coefficient of the insulating plates 60.

[0070] For this reason, during conventional welding (e.g., brazing) of an alumina insulating plate and an aluminum frame (the first frame part), when the insulating plate and the first frame part are heated and then cooled, the insulator and the frame are liable to warp (bend) due to a difference in shrinkage rate (linear expansion coefficient) between the insulator and the first frame part.



[0071] In Example 1, on the other hand, the plurality of protrusions (four protrusions in Example 1) **31b**, **31c**, **31d**, and **31e** each protruding into the frame **30** are integrally formed with the first frame part **31** as mentioned above (see FIG. 2). Each of those protrusions **31b** to **31e** has the U-shaped groove configuration having the U-shaped cross section that opens outward of the frame **30** (upward in FIG. 5) and linearly extending in the direction B along the mounting surfaces **31g**. Those protrusions **31b** to **31e** configured as above are elastically deformable in a direction along the mounting surfaces **31g** of the first frame part **31** (in a lateral direction in FIGS. 5 and 11) by widening and narrowing the openings of the U-shaped grooves (see FIG. 11).

[0072] Consequently, when each insulating plate **60** is welded (brazed in Example 1) to the first frame part **31**, the protrusions **31b** to **31e** are elastically deformed in the direction (in the lateral direction in FIG. 11) along the mounting surfaces **31g** of the first frame part **31** according to the difference in shrinkage rate (linear expansion coefficient) between the insulating plates **60** and the first frame part **31**. This can prevent the warping (bending) of the insulating plates **60** and the first frame part **31** due to the difference in their shrinkage rate (linear expansion coefficient). In Example 1, the protrusions **31b** to **31e** correspond to an elastically deformable portion.

[0073] Moreover, a heat exchanger including a fin member provided with a plurality of fins thereby forming coolant flow channels in a frame constituting an outer casing was subjected to an investigation to find a distribution of velocity of the coolant flowing between the fins. This investigation showed the tendency that the velocity of the coolant flowing nearer the fins was slower. This is because the coolant is attracted by the fins due to the influence of viscosity of the coolant. Accordingly, near the fins, there occurs a region in which the flowing velocity of the coolant is slower than in other regions or the coolant hardly flows (hereinafter, this region is also referred to as a boundary layer). If such a boundary layer occurs, the fins that collect heat will exchange heat with only the coolant in the boundary layer generated mainly around the fins and will hardly exchange heat with the coolant flowing in other regions than the boundary layer. As a result, heat exchange with the coolant flowing in the heat exchanger can not be effectively conducted and a high cooling effect could not be achieved.

[0074] In contrast, the heat exchanger **10** in Example 1 is configured such that the protrusions **31b** to **31e** each protruding into the frame **30** extend linearly in the direction (direction B) perpendicular to the coolant flowing direction (see FIG. 2). This configuration can make the coolant flowing in the flow channels **25** collide with the protrusions **31b** to **31e** and agitate the coolant as shown in FIG. 7. Therefore, as shown by an arrow in FIG. 7, the flow of coolant is brought into a turbulent state, thereby effectively preventing the occurrence of the boundary layer. Consequently, the coolant flowing in the heat exchanger **10** can be effectively utilized to provide a high cooling effect.

[0075] In the heat exchanger **10** in Example 1, furthermore, the protrusions **31b** to **31e** are provided to extend to inner wall surfaces **33b** and **34b** of both side walls **33** and **34** of the second frame part **32** in the direction B (perpendicular to the coolant flowing direction) (see FIG. 2). Thus, the coolant flowing in the flow channels **25** formed between the fins **22** adjacently located in the direction B, between the fins **22** and the side wall **33**, and between the fins **22** and the side wall **34**

can be made reliably collide with the protrusions **31b** to **31e** and agitated. Accordingly, the coolant flowing in all the flow channels **25** is surely made turbulent so that the occurrence of the boundary layer is prevented more effectively. Consequently, the coolant flowing in the heat exchanger **10** can be more effectively utilized to achieve a higher cooling effect.

[0076] In the heat exchanger **10** in Example 1, furthermore, the protrusions **31b** to **31e** are arranged side by side (in parallel) and at predetermined intervals in the direction A (the coolant flowing direction) and the fins **22** are placed between adjacent protrusions in the direction A (the coolant flowing direction). Since the fins **22** are placed between the protrusions adjacently located in the coolant flowing direction (the direction A), the coolant flowing in the flow channel **25** formed between the fins **22** can be made to appropriately collide with the protrusions and be agitated. This makes it possible to prevent the occurrence of the boundary layer and achieve a higher cooling effect.

[0077] Here, a cooling operation of the heat exchanger **10** of the semiconductor device **1** in Example 1 is explained. The semiconductor elements **71** to **74** generate heat as they operate. Such heat is transferred to the frame **30** (the first frame part **31**) through the insulating plates **60** and further to the fins **22** of the fin member **20** accommodated in the frame **30**.

[0078] The coolant (e.g., water) is continuously introduced into the frame **30** through the inlet **30a** as indicated by a thick arrow in FIG. 1. The coolant introduced into the frame **30** is allowed to flow, in the direction A, in the flow channels **25** formed between the fins **22** adjacently located in the direction B and between the fins **22** and the side wall **33** or **34** as mentioned above. The coolant flowing forward in the flow channels **25** collides with the protrusions **31b** to **31e** and thus becomes agitated as described above. That is, the coolant flows forward in the direction A while making turbulent flows (see FIG. 7). Such a flow can prevent the occurrence of the boundary layer.

[0079] Accordingly, the fins **22** of the fin member **20** can make effective heat exchange with the coolant flowing in the flow channels **25**. In other words, the heat transferred from the semiconductor elements **71** to **74** to each fin **22** can be dissipated into the coolant flowing in the flow channels **25**. The high cooling effect achieved by the heat exchanger **10** is as mentioned above. The coolant having absorbed the heat from the fins **22** as flowing in the flow channels **25** is discharged out of the frame **30** through the outlet **30b**. In such a way, the semiconductor elements **71** to **74** that generate heat can be effectively cooled.

[0080] A method for manufacturing the semiconductor device **1** in Example 1 will be explained below.

[0081] The fin member **20** made of aluminum is first prepared. This fin member **20** can be made of aluminum into a single piece by extrusion for example. To be concrete, a long fin member is formed of aluminum by extrusion to include a rectangular flat plate-like base and a plurality of long fins (ten long fins in Example 1) each protruding from one surface of the base and extending in a longitudinal direction (corresponding to the direction A) of the base. Just after the extrusion but before the long fin member is hardened (in a hot and soft state), a part of the long fin is cut at predetermined intervals in the longitudinal direction (corresponding to the direction A) of the base. Thereafter, the long fin member is cut into a predetermined length, providing a complete fin member **20** including the rectangular flat plate-like base **21** and the plurality of fins (forty fins in Example 1) **22** protruding from



one surface of the base **21** (see FIG. 3). The details of this manufacturing method are disclosed in Japanese Patent Application No. 2008-106809 (see FIG. 5).

[0082] The first frame part **31** made of aluminum and the second frame part **32** made of aluminum are further prepared.

[0083] The first frame part **31** is formed as below in a forming process using a first press die **91**, a second press die **92**, and a drawing die **93** (see FIG. 8). The drawing die **93** has a rectangular flat plate-like shape. A size of the die **93** in its longitudinal direction (a direction perpendicular to a drawing sheet of FIG. 8, corresponding to the direction B) is slightly shorter than the protrusions **31b** to **31e** in the direction B. The first press die **91** is formed with a through hole **91b** in which the drawing die **93** is insertable. The second press die **92** is formed with a through hole **92b** having a larger width (in a lateral direction in FIG. 8) than a width of the through hole **91b**.

[0084] The rectangular aluminum flat plate **31A** is first prepared. Successively, this aluminum plate **31A** is held between and fixed by the first press die **91** disposed on an upper surface of the aluminum plate **31A** and the second press die **92** disposed on a lower surface of the plate **31A** as shown in FIG. 8. In this state, the drawing die **93** is moved downward from above the upper surface of the aluminum plate **31A** and inserted into the through hole **91b** of the first press die **91**, gradually pressing down a part of the plate **31A**. The part of the plate **31A** pressed downward by the drawing die **93** is pushed out of the second press die **92** through the through hole **92b** of the second press die **92**. Thus, a protrusion **31b** is formed. The protrusions **31c** to **31e** are also formed in the same manner. As above, the protrusions **31b** to **31e** are made by drawing. Consequently, the first frame part **31** integrally formed with the protrusions **31b** to **31e** is completed.

[0085] The second frame part **32** can be manufactured in such a manner that a rectangular aluminum flat plate is formed into a rectangular U-shape by pressing.

[0086] In a subsequent assembling process, the fin member **20** is put on an inner surface **31h** of the first frame part **31** as shown in FIG. 9. To be concrete, the fin member **20** is placed on the inner surface **31h** of the first frame part **31** so that the protrusions **31c** to **31e** of the first frame part **31** enter in clearances between the fins **22** of the fin member **20**, located adjacently in the direction A. Thereafter, the second frame part **32** is placed on the inner surface **31h** of the first frame part **31** to surround the fin member **20**. At that time, a back surface **21b** of the base **21** of the fin member **20** comes into contact with a bottom surface **32b** of the second frame part **32**. The inner surface **31h** of the first frame part **31** and the bottom surface **32b** of the second frame part **32** are applied in advance with brazing filler metal (Melting point: 600° C.).

[0087] Thereafter, as mentioned above, an assembly of the fin member **20**, first frame part **31**, and second frame part **32** is put in an electric furnace (not shown). The temperature of this furnace is then increased to 600° C. to melt the brazing filler metal. Subsequently, the assembly is taken out of the furnace and then cooled to harden the brazing filler metal. The fin member **20**, the first frame part **31**, and the second frame part **32** can thus be joined by brazing. As above, the heat exchanger **10** in Example 1 can be completed.

[0088] In a welding process, four insulating plates **60** are welded (brazed in Example 1) to the first frame part **31** of the heat exchanger **10**. To be specific, four insulating plates **60** (e.g., ceramic plates made of alumina) are arranged in a line at equal intervals (by interposing the protrusions **31c** to **31e**

one between each two of the insulating plates **60**) in the longitudinal direction (corresponding to the direction A) of the first frame part **31** on the mounting surfaces **31g** of the first frame part **31** of the heat exchanger **10** (see FIG. 10). The mounting surfaces **31g** of the first frame part **31** are applied with brazing filler metal (Melting point: 550° C.) after the heat exchanger **10** is completed but before the insulating plates **60** are mounted. After they are put in the electric furnace (not shown), the temperature of the furnace is increased up to 550° C. to melt the brazing filler metal. At that time, the brazing filler metal joining the heat exchanger **10** has a melting point of 600° C. and therefore does not melt. The joining state of the heat exchanger **10** can be surely maintained. Accordingly, the heat exchanger **10** provided with the insulating plates **60** is taken out of the electric furnace and cooled to harden the brazing filler metal. As above, four insulating plates **60** can be brazed to the first frame part **31** of the heat exchanger **10**.

[0089] In addition, the insulating plates **60** and the frame **30** (the first frame part **31**) are different in linear expansion coefficient. In the case where the insulating plates **60** made of alumina, for instance, the linear expansion coefficient of the frame **30** (the first frame part **31**) made of aluminum is more than three times as high as the linear expansion coefficient of the insulating plates **60**. During welding (e.g., brazing) of an alumina insulating plate and an aluminum frame (the first frame part), conventionally, when the insulating plate and the first frame part are heated and then cooled, the insulator and the frame are liable to warp (bend) due to a difference in shrinkage rate (linear expansion coefficient) between the insulator and the first frame part.

[0090] In Example 1, in contrast, the plurality of protrusions (four protrusions in Example 1) **31b**, **31c**, **31d**, and **31e** protruding into the frame **30** are integrally formed with the first frame part **31** (see FIG. 2). Each of those protrusions **31b** to **31e** has the U-shaped groove configuration having the U-shaped cross section that opens outward (upward in FIG. 5) of the frame **30** and linearly extending in the direction B along the mounting surfaces **31g**. By widening and narrowing the openings of the U-shaped grooves, the protrusions **31b** to **31e** configured as above are elastically deformable in the direction (in the lateral direction in FIGS. 5 and 11) along the mounting surfaces **31g** of the first frame part **31** (see FIG. 11).

[0091] During welding (brazing in Example 1) of the insulating plates **60** and the first frame part **31**, when the insulating plates **60** and the first frame part **31** are heated and then cooled, the protrusions **31b** to **31e** are elastically deformed (shrinkage) in the direction (in the lateral direction in FIG. 11) along the mounting surfaces **31g** of the first frame part **31** according to a difference in shrinkage rate (linear expansion coefficient) between the insulating plates **60** and the first frame part **31**. This makes it possible to prevent the insulating plates **60** and the first frame part **31** from warping (bending) due to the difference in shrinkage rate (linear expansion coefficient) between the insulating plates **60** and the first frame part **31**. Then, the semiconductor elements **71** to **74** are respectively soldered to the surfaces of the insulating plates **60**. The semiconductor device **1** (see FIGS. 1 and 6) in Example 1 is thus completed.

#### Example 2

[0092] Example 2 of the present invention will be explained referring to the accompanying drawings.



[0093] A semiconductor device **100** (see FIG. **12**) in Example 2 is identical to the semiconductor device **1** (see FIG. **1**) in Example 1 excepting a heat exchanger alone. Accordingly, only different configurations from those in Example 1 are mainly explained below and the same or identical parts are not explained or are briefly mentioned.

[0094] A heat exchanger **110** in Example 2 includes a frame **130** forming an outer casing and fin members **120** and **220** accommodated in the frame **130** as shown in FIG. **13**. The frame **130** is joined to the fin members **120** and **220** by brazing. A direction A represents a flowing direction of a coolant (e.g., water) allowed to flow in the heat exchanger **110** and a direction B represents a direction perpendicular to the direction A.

[0095] The frame **130** includes a first frame part **131** made of aluminum in a rectangular flat plate-like shape and a second frame part **132** made of aluminum in a rectangular U-shaped cross section (see FIG. **13**). The first frame part **131** and the second frame part **132** are joined to each other by brazing.

[0096] The first frame part **131** includes four protrusions (see FIGS. **16** and **17**) as with the first frame part **31** in Example 1. However, two of the four protrusions are different in shape from the protrusions in Example 1. To be concrete, the first frame part **131** includes protrusions **131b** and **131c** having the same shape of the protrusions **31b** and **31c** in Example 1 and protrusions **131d** and **131e** different from the protrusions **31d** and **31e** in Example 1 only in shorter protruding height. Those protrusions **131b** to **131e** are integrally formed in the first frame part **131** by drawing as in Example 1. In FIG. **17**, the protruding height (length) of each protrusion **131b** and **131c** is referred to as L1 and the protruding height (length) of each protrusion **131d** and **131e** is referred to as L2.

[0097] Further, the second frame part **132** is different from the second frame part **32** in Example 1 in that the second frame part **132** is deeper (a size in a vertical direction in FIG. **13**) on a downstream side of the flow channels **125** extending in the direction A (in the coolant flowing direction) than the second frame part **32**. In other words, a bottom surface **132b** of the second frame part **132** is stepped downward in the direction A. Accordingly, the flow channel depth (the size in the vertical direction in FIG. **13**) of the flow channels **125** extending in the direction A (the coolant flowing direction) is deeper on the downstream side than on the upstream side. Specifically, as shown in FIG. **19**, each flow channel **125** has a depth H1 on the upstream side and a depth H2 on the downstream side so that the depth H2 is larger than the depth H1.

[0098] The flow channels **125** are formed between the fins **22** adjacent in the direction B, between the fins **22** and the side wall **133**, between the fins **22** and the side wall **134**, between fins **222** adjacent in the direction B, between the fins **222** and the side wall **133**, and between the fins **222** and the side **134**.

[0099] The fin members **120** (see FIG. **14**) are members each corresponding to one piece obtained by dividing the fin member **20** in Example 1 into four in the direction A. To be specific, each fin member **120** is made of aluminum including a rectangular flat plate-like base **121** and a plurality of fins (ten fins in Example 1) **22** protruding from one surface of the base **121**. These fin members **120** are placed one each between the protrusions **131b** and **131c** of the first frame part **131** and between the protrusions **131c** and **131d** (see FIG. **13**). The fin members **120** are thus placed on the upstream side of

the flow channels **125**. In addition, the flow channels **125** each having a predetermined width are formed between the fins **22** to introduce the coolant in the direction A. Each fin member **120** can be formed of an integral component by known extrusion for example.

[0100] Each fin member **220** is made of aluminum and includes a rectangular flat plate-like base **121** and a plurality of fins (ten fins in Example 1) **222** protruding from one surface of the base **121** as shown in FIG. **15**. These fin members **220** are placed one each between the protrusions **131d** and **131e** of the first frame part **131** and in a forward adjacent position to the protrusion **131e** in the direction A (see FIG. **13**). Accordingly, the fin members **220** are located on a downstream side of the flow channels **125**. Each fin member **220** is different only in fin protruding height (a size in a vertical direction in FIG. **15**) from the fin members **120**. Specifically, each fin member **220** is designed such that the fins **222** have a protruding height higher than the protruding height of the fins **22** of each fin member **120** by a length (H2-H1) determined by subtracting the depth H1 of the flow channels **125** on the upstream side from the depth H2 on the downstream side. In addition, the flow channels **125** each having a predetermined width are formed between the fins **222** to introduce the coolant in the direction A. Each fin member **120** can be formed of an integral component by known extrusion for example.

[0101] Moreover, as the protruding height of the protrusions of the first frame part is higher, the flow resistance of coolant flowing in the flow channels can be made higher. Thus, the protrusions can provide the higher ability to agitate the coolant flowing in the flow channels, thereby enhancing the effect of preventing the occurrence of the boundary layer. However, if the protruding height of the protrusions is too high, pressure loss in the flow channels becomes too larger. This greatly lowers the flow amount of coolant and thus results in a low cooling effect.

[0102] When a plurality of semiconductor elements having different heat generation amounts are to be cooled, it is unnecessary to cool all the heat generating bodies uniformly and the semiconductor elements have only to be cooled according to respective heat generation amounts. Consequently, of the semiconductor elements, the semiconductor element(s) having a relatively low heat generation amount may be cooled with a lower degree of cooling than the other semiconductor element(s) is.

[0103] In the semiconductor device **100** in Example 2, among the semiconductor elements **71** to **74**, the semiconductor elements **73** and **74** have lower heat generation amounts than the semiconductor elements **71** and **72** have. In the heat exchanger **110** in Example 2, therefore, the protruding height L2 of the protrusions **131d** and **131e** arranged respectively upstream (a right side in FIG. **19**) in the flow channels **125** from and adjacent to low-heat generation portions **131k** is shorter than the protruding height L1 of the other protrusions **131b** and **131c**. Here, the low-heat generation portions **131k** are portions of the first frame part **131** and on which the semiconductor elements **73** and **74** having relatively low heat generation amounts are placed through the insulating plates **60**. Since the protruding height of the selected protrusions **131d** and **131e** is designed to be shorter, pressure loss in the flow channels **125** can be reduced.

[0104] On the other hand, the protrusions **131d** and **131e** with shorter protruding height are lower in coolant agitating ability than the other protrusions **131b** and **131c**. In the flow channel sections **125d** and **125e** arranged respectively down-



stream (a left side in FIG. 19) in the flow channels **125** from and adjacent to the protrusions **131d** and **131e** with shorter protruding height, the effect of preventing the occurrence of the boundary layer decreases. In Example 2, however, the protrusions **131d** and **131e** arranged respectively upstream (the right side in FIG. 19) in the flow channel **125** from and adjacent to the low-heat generation portions **131k** are designed with shorter protruding height. In other words, the first frame part **131** is configured to include portions arranged downstream (the left side in FIG. 19) in the flow channel **125** from and adjacent to the protrusions **131d** and **131e** with shorter protruding height as the low-heat generation portions **131k**. Accordingly, only the low-heat generation portions **131k** have the low effect of preventing the occurrence of the boundary layer and the low cooling ability. In this way, the cooling ability is low in only the low-heat generation portions **131k**, so that the semiconductor elements **71** to **74** can be cooled appropriately according to respective heat generation amounts.

[0105] In the heat exchanger **110** in Example 2, as shown in FIG. 19, the flow channel sections **125d** and **125e** including the low-heat generation portions **131k** in each coolant flow channel **125** are designed with a deeper depth than the depth of other flow channel sections (the flow channel sections **125b** and **125c** including high-heat generation portions **131j**). The flow resistance of coolant flowing in the flow channels **125** can be reduced by the larger depth of the flow channel sections **125d** and **125e**. the pressure loss in the flow channels **125** can be further reduced. In FIG. 19, the depth of each flow channel section **125b** and **125c** is referred to as H1 and the depth of each flow channel section **125d** and **125e** is referred to as H2. The high-heat generation portions **131j** are portions of the first frame part **131** and on which the semiconductor elements **71** and **72** having relatively large heat generation amounts are placed through the insulating plates **60**.

[0106] Meanwhile, in a flow channel section with deeper depth, the flow velocity tends to become faster than in other flow channel section, resulting in a lower cooling ability. In the heat exchanger **110** in Example 2, however, only the flow channel sections **125d** and **125e** including the low-heat generation portions **131k** are designed with deeper depth. Since the cooling ability is low in only the low-heat generation portions **131k**, the semiconductor elements **71** to **74** can be cooled appropriately according to respective heat generation amounts.

[0107] Here, a cooling operation of the heat exchanger **110** of the semiconductor device **100** in Example 2 is explained. The semiconductor elements **71** to **74** generate heat as they operate. Such heat is transferred to the frame **130** (the first frame part **131**) through the insulating plates **60** and further to the fins **22** of the fin members **120** and the fins **222** of the fin members **220** accommodated in the frame **130**.

[0108] The coolant (e.g., water) is continuously introduced into the frame **130** through the inlet **130a** as indicated by a thick arrow in FIG. 12. The coolant introduced into the frame **130** is allowed to flow in the flow channels **125** in the direction A. The coolant flowing in the flow channels **125** collide with the protrusions **131b** to **131e** and thus becomes agitated, so that the coolant flows forwards in the direction A while making turbulent flows (see FIG. 19). Such a flow can prevent the occurrence of the boundary layer.

[0109] Accordingly, the fins **22** of the fin members **120** and the fins **222** of the fin members **220** can make effective heat exchange with the coolant flowing in the flow channels **125**.

Specifically, the heat transferred from the semiconductor elements **71** to **74** to the fins **22** and **222** can be efficiently dissipated into the coolant flowing in the flow channels **125**. The coolant having absorbed the heat from the fins **22** and **222** while flowing in the flow channels **125** is discharged out of the frame **130** through the outlet **130b**. In such a way, the semiconductor elements **71** to **74** that generate heat can be effectively cooled.

[0110] In addition, the heat exchanger **110** in Example 2 is configured such that the protrusions **131d** and **131e** are of shorter protruding height and the flow channel sections **125d** and **125e** are of deeper depth as mentioned above. This configuration can effectively reduce the pressure loss in the flow channels **125**. It is therefore possible to restrain a decrease in flow amount of coolant, thereby enabling appropriate cooling of the semiconductor elements **71** to **74** that generate heat.

[0111] A method for manufacturing semiconductor device **100** in Example 2 will be explained below.

[0112] The fin members **120** and **220** made of aluminum are first prepared two by two. Each fin member **120** and **220** can be made of aluminum into a single piece by extrusion for example.

[0113] Further, the first frame part **131** made of aluminum and the second frame part **132** made of aluminum are also prepared. The first frame part **131** is fabricated in a forming process using the first press die **91**, the second press die **92**, and the drawing die **93** (see FIG. 8) in the same manner as in Example 1. However, in the drawing work for the protrusions **131d** and **131e**, a downward moving amount (a drawing amount) of the drawing die **93** is set smaller than in the drawing work for the protrusions **131b** and **131c**. The second frame part **132** can be manufactured from a rectangular aluminum flat plate by pressing.

[0114] In an assembling process, subsequently, the fin members **120** and **220**, the first frame part **131**, and the second frame part **132** are assembled together as shown in FIG. 13 (constituting an assembly). An inner surface **131h** of the first frame part **131** and a bottom surface **132b** of the second frame part **132** are applied in advance with brazing filler metal (Melting point: 600° C.). Thereafter, as in Example 1, the assembly is put in the electric furnace (not shown). The brazing filler metal is melted therein and then the assembly is cooled to harden the brazing filler metal. Thus, the fin members **120** and **220**, the first frame part **131**, and the second frame part **132** are joined to each other by brazing. In such a way, the heat exchanger **110** in Example 2 is completed.

[0115] In a welding process, four insulating plates **60** are brazed to the first frame part **131** of the heat exchanger **110** according to the same procedures as in Example 1. Specifically, the four insulating plates **60** are first placed one on each of the mounting surfaces **131g** (see FIGS. 16 and 17) of the first frame part **131**. It is to be noted that the mounting surfaces **131g** of the first frame part **131** are applied with brazing filler metal (Melting point: 550° C.) after the heat exchanger **110** is completed but before the insulating plates **60** are mounted. Subsequently, they are put in the electric furnace (not shown) and then the temperature of the furnace is increased to 550° C. to melt the brazing filler metal. At that time, the brazing filler metal joining the heat exchanger **110** has a melting point of 600° C. and therefore does not melt. Accordingly, the joining state of the heat exchanger **110** can be surely maintained. After that, the heat exchanger **110** with the insulating plates **60** is taken out of the furnace and cooled,



thus hardening the brazing filler metal. In such a way, the insulating plates **60** are brazed to the first frame part **131** of the heat exchanger **110**.

[0116] Meanwhile, the insulating plates **60** and the frame **130** (the first frame part **131**) are different in linear expansion coefficient. In the case where the alumina insulating plates **60** are used, for example, the linear expansion coefficient of the aluminum frame **130** (first frame part **131**) is more than three times as high as the linear expansion coefficient of the insulating plates **60**. Therefore, during welding (e.g., brazing) of an alumina insulating plate and an aluminum frame (first frame part), conventionally, when the insulating plate and the first frame part are heated and then cooled, the insulator and the frame are liable to warp (bend) due to a difference in shrinkage rate (linear expansion coefficient) between them.

[0117] In Example 2, in contrast, a plurality of protrusions (four protrusions in Example 1) **131b**, **131c**, **131d**, and **131e** protruding into the frame **130** are integrally formed in the first frame part **131** as mentioned above (see FIGS. **13** and **17**). Each of those protrusions **131b** to **131e** has a U-shaped groove configuration having a U-shaped cross section that opens outward (upward in FIG. **17**) of the frame **130** and linearly extending in the direction B along the mounting surfaces **131g**. Those protrusions **131b** to **131e** configured as above are elastically deformable in the direction along the mounting surfaces **131g** of the first frame part **131** (in a lateral direction in FIGS. **11** and **17**) by widening and narrowing the openings of the U-shaped grooves (see FIG. **11**).

[0118] Consequently, during welding (brazing in Example 2) of each insulating plate **60** to the first frame part **131**, when the insulating plates **60** and the first frame part **131** are heated and then cooled, the protrusions **131b** to **131e** are elastically deformed in the direction (in the lateral direction in FIG. **11**) along the mounting surfaces **131g** of the first frame part **131** according to the difference in shrinkage rate (linear expansion coefficient) between the insulating plates **60** and the first frame part **131**. This can prevent the insulating plates **60** and the first frame part **131** from warping (bending) due to the difference in their shrinkage rate (linear expansion coefficient).

[0119] Thereafter, the semiconductor elements **71** to **74** are soldered respectively to the surfaces of the insulating plates **60**. The semiconductor device **100** in Example 1 is thus completed (see FIGS. **12** and **18**).

[0120] The present invention is explained above in Examples 1 and 2 but is not limited thereto. The present invention may be embodied in other specific forms without departing from the essential characteristics thereof.

[0121] For instance, in Example 1, the fin member **20**, the first frame part **31**, and the second frame part **32** are first joined to each other by brazing to complete the heat exchanger **10**, and subsequently the insulating plates **60** are brazed to the first frame part **31** of the heat exchanger **10**. As an alternative, the fin member **20**, the first frame part **31**, the second frame part **32**, and the insulating plates **60** may be brazed to each other at the same time. In this case, as the brazing filler metal, metals having the same melting point (e.g., 600° C.) may be used. According this method, at the same time when the heat exchanger **10** is completed, the insulating plates **60** are brazed to the first frame part **31** of the heat exchanger **10**. The same applies to Example 2.

#### DESCRIPTION OF THE REFERENCE SIGNS

[0122] **1**, **100** Semiconductor device  
[0123] **10**, **110** Heat exchanger

[0124] **20**, **120**, **220** Fin member  
[0125] **22**, **222** Fin  
[0126] **25**, **125** Flow channel  
[0127] **30**, **130** Frame  
[0128] **31**, **131** First frame part (First wall part)  
[0129] **31b**, **31c**, **31d**, **31e**, **131b**, **131c**, **131d**, **131e** Protrusion (Elastically deformable portion)  
[0130] **31f**, **131f** Outer surface of First frame part (First wall part)  
[0131] **31g**, **131g** Mounting surface  
[0132] **32**, **132** Second frame part  
[0133] **60** Insulating plate (Intervening member)  
[0134] **71**, **72**, **73**, **74** Semiconductor device (Heat generating body)  
[0135] **125d**, **125e** Flow channel section including a low-heat generation portion  
[0136] **131k** Low-heat generation portion  
[0137] A Coolant flowing direction  
[0138] B Direction perpendicular to (intersecting with) coolant flowing direction

1. (canceled)

2. A heat exchanger for cooling a heat generating body, the heat exchanger including a frame forming an outer casing and a fin member having a plurality of fins forming flow channels for a coolant, the fin member being placed in the frame, wherein

the frame includes a first wall part to which an intervening member is welded so that the intervening member is interposed between the frame and the heat generating body, the intervening member having a different linear expansion coefficient from a linear expansion coefficient of the frame,

the first wall part of the frame includes an elastically deformable portion configured to be deformable in a direction along a mounting surface of an outer surface of the first wall part, on which mounting surface the intervening member is placed, and

the elastically deformable portion includes a protrusion protruding into the frame, the protrusion having a U-shaped groove configuration having a U-shaped cross section that opens outward of the frame and linearly extending along the mounting surface in a direction intersecting with a coolant flowing direction.

3. The heat exchanger according to claim 2, wherein the protrusion is integrally formed in the wall part by drawing.

4. The heat exchanger according to claim 2, wherein the protrusion is provided to extend to inner wall surfaces of both side walls of the frame in the direction intersecting with the coolant flowing direction.

5. The heat exchanger according to claim 2, wherein the protrusion linearly extends in a direction perpendicular to the coolant flowing direction.

6. The heat exchanger according to claim 2, wherein the protrusion includes protrusions arranged side by side at predetermined intervals in the coolant flowing direction, and the fins are placed between the protrusions located adjacently in the coolant flowing direction.

7. The heat exchanger according to claim 6, wherein the heat exchanger is configured to cool a plurality of heat generating bodies arranged side by side in the coolant flowing direction,

the first wall part includes a low-heat generation portion on which a heat generating body having a relatively small



heat generation amount among the heat generating bodies is placed through the intervening member, and of the protrusions, a protrusion located upstream in the flow channel from and adjacent to the low-heat generation portion has a shorter protruding height protruding into the frame than a protruding height of other protrusion.

**8.** The heat exchanger according to claim 7, wherein a flow channel section of the coolant flow channel, the flow channel section including the low-heat generation portion, is deeper in flow channel depth than other flow channel sections are.

**9.** A semiconductor device including:  
the heat exchanger according to claim 2;  
a semiconductor element that is the heat generating body;  
and  
the intervening member welded to the first wall part of the frame and located between the semiconductor element and the first wall part.

**10.** A method for manufacturing a heat exchanger for cooling a heat generating body, the heat exchanger including a frame forming an outer casing and a fin member having a plurality of fins forming flow channels for coolant, the fin member being placed in the frame, the method comprising:

a forming step of forming the frame including a first wall part to which an intervening member to be interposed between the frame and the heat generating body is welded; and

an assembling step of assembling the heat exchanger by placing the fin member in the frame formed in the forming step,

the forming step including drawing to form the protrusion protruding into the frame integral with the first wall part so that the protrusion has a U-shaped groove configuration having a U-shaped cross section that opens outward of the frame and linearly extending along a mounting surface on which the intervening member is to be placed, of the outer surface of the first wall part.

**11.** A method for manufacturing a semiconductor device comprising:

the heat exchanger manufactured by the heat exchanger manufacturing method according to claim 10;  
a semiconductor element that is the heat generating body;  
and

an intervening member to be interposed between the first wall part of the frame and the semiconductor element, the intervening member having a different linear expansion coefficient from a linear expansion coefficient of the frame, the method comprising:

a welding step of placing the intervening member on the mounting surface of the first wall part of the frame and then welding the intervening member to the first wall part.

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