

US 20090191073A1

(19) United States

(12) Patent Application Publication Kopecek et al.

(10) Pub. No.: US 2009/0191073 A1 (43) Pub. Date: Jul. 30, 2009

(54) MAGNETIC PUMPING MACHINES

(75) Inventors: Herbert Kopec

Herbert Kopecek, Hallbergmoos (DE); Michael Bernhard Schmitz, Freising (DE); Johannes Eckstein, Ismaning (DE); Nicola Marcucci, Florence (IT); Stefano Meucci, Florence (IT); Carlos Jimenez Haertel, Muenchen (DE)

Correspondence Address:

GENERAL ELECTRIC COMPANY GLOBAL RESEARCH PATENT DOCKET RM. BLDG. K1-4A59 NISKAYUNA, NY 12309 (US)

(73) Assignee: GENERAL ELECTRIC

COMPANY, Schenectady, NY

(US)

(21) Appl. No.: 12/019,697

(22) Filed: **Jan. 25, 2008**

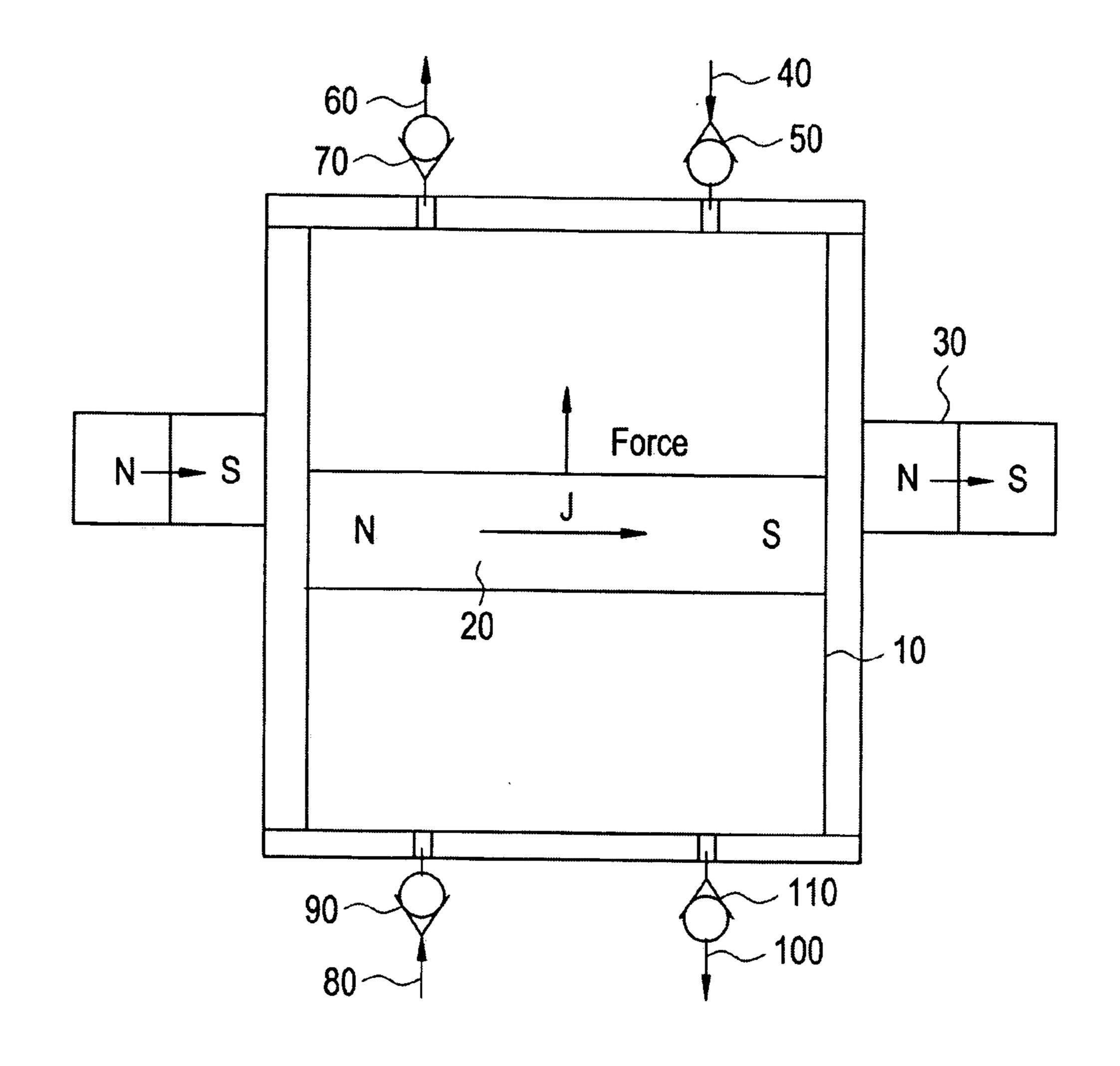
Publication Classification

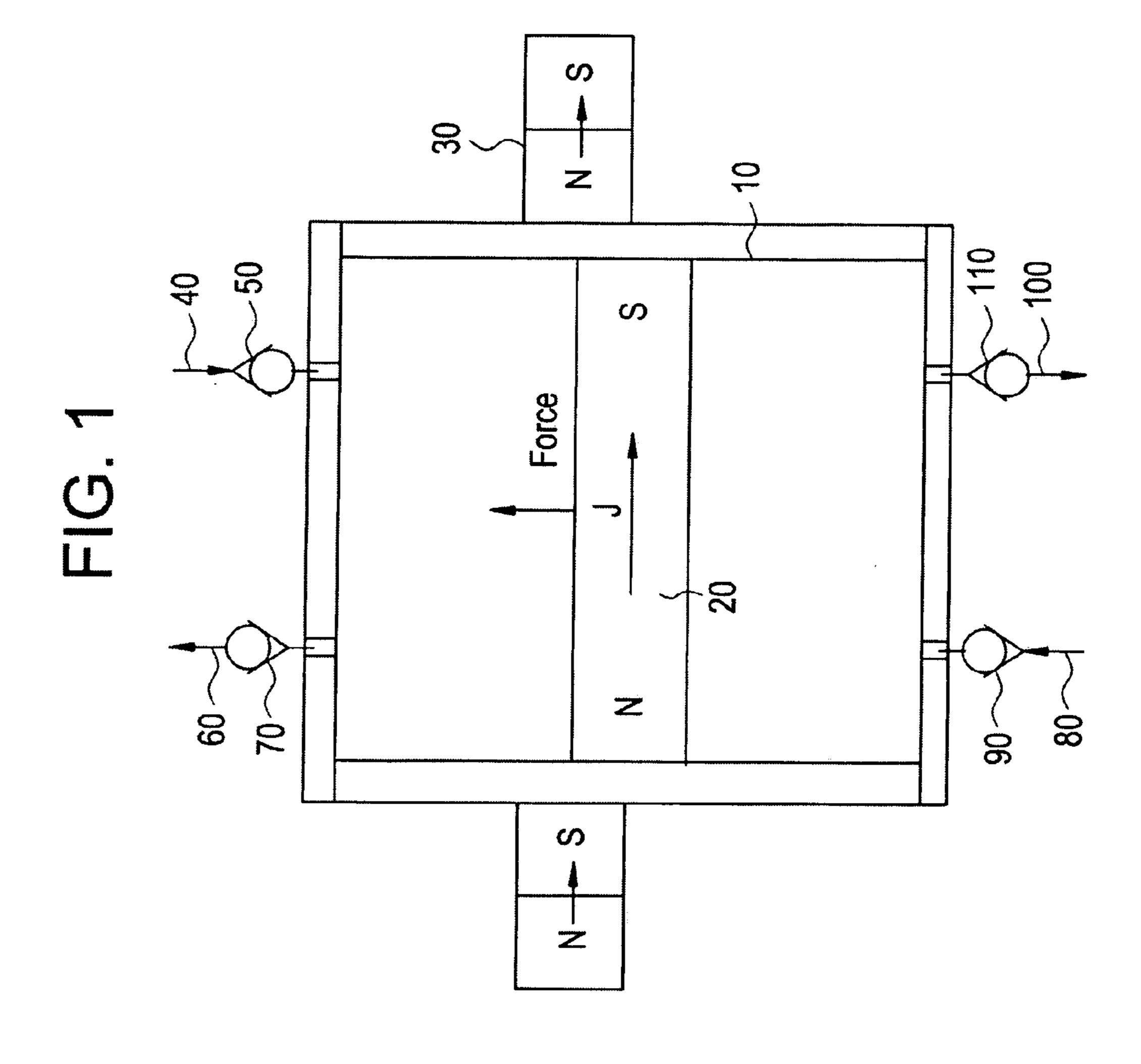
(51) Int. Cl. F04B 17/00 (2006.01) F04B 35/04 (2006.01)

(2) U.S. Cl. 417/41

(57) ABSTRACT

Magnetic pumping systems and methods of pumping using such systems are disclosed. In one embodiment, a system for pumping fluid comprises: a substantially cylinder-shaped pump housing (200); a magnetic piston (210) residing within the pump housing (200) for displacing the fluid; a magnetic actuator (220) surrounding the pump housing (200), wherein the magnetic actuator (220) is configured to move the piston (210) by a magnetic force between the magnetic piston (210) and the magnetic actuator (220); and a magnetic ring (230) surrounding and attached to the magnetic actuator (220) for increasing a magnitude of the magnetic force. In another embodiment, a system for pumping fluid comprises: a pump housing (10); a magnetic piston (20) residing within the pump housing (10) for displacing the fluid; an electromagnetic actuator (150) outside the pump housing (10) comprising a movable conductive coil (160), wherein the electromagnetic actuator (150) is configured to move the piston (20) by an electromagnetic force; and an external driver for moving the conductive coil (160).





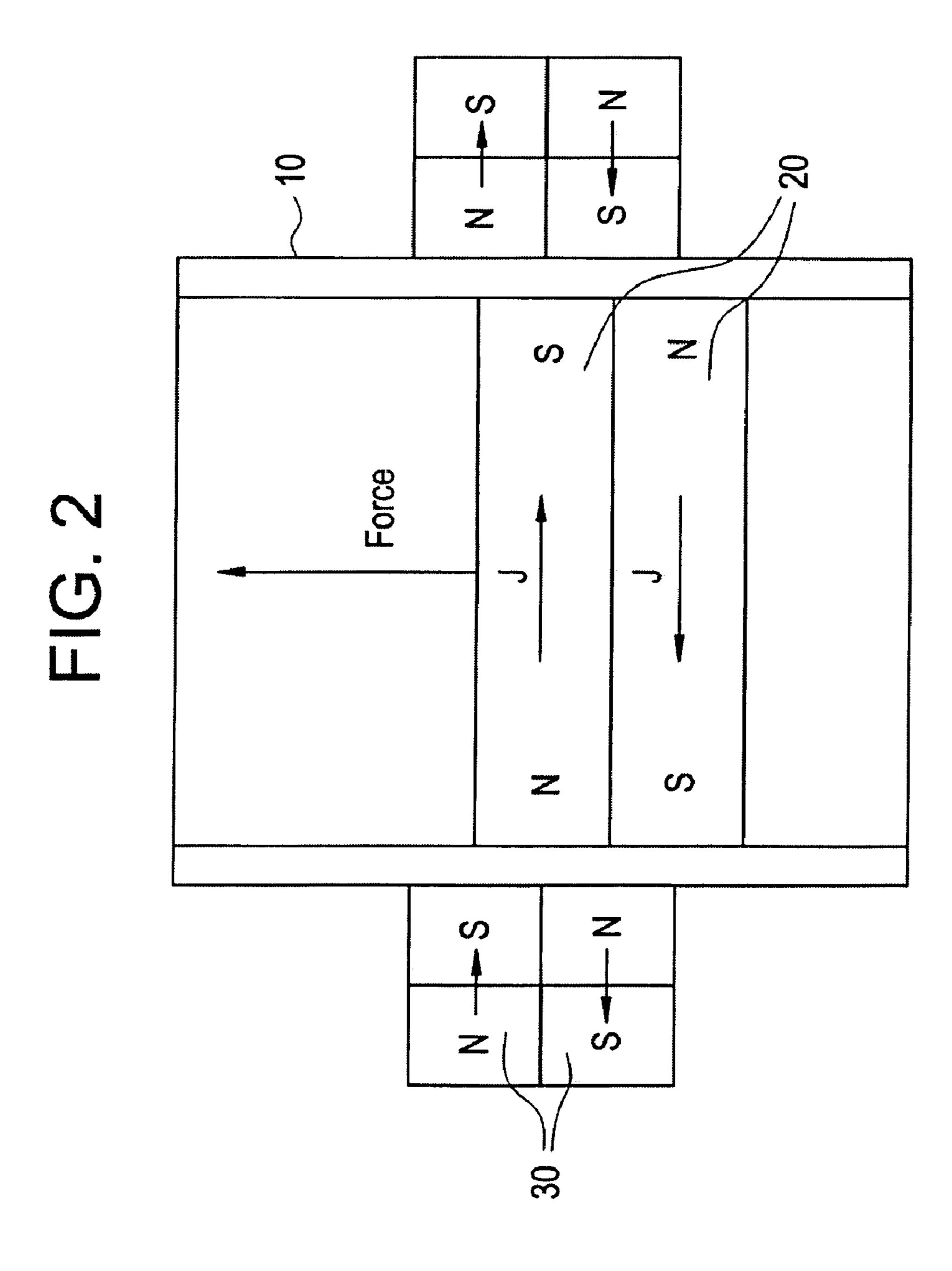


FIG. 3

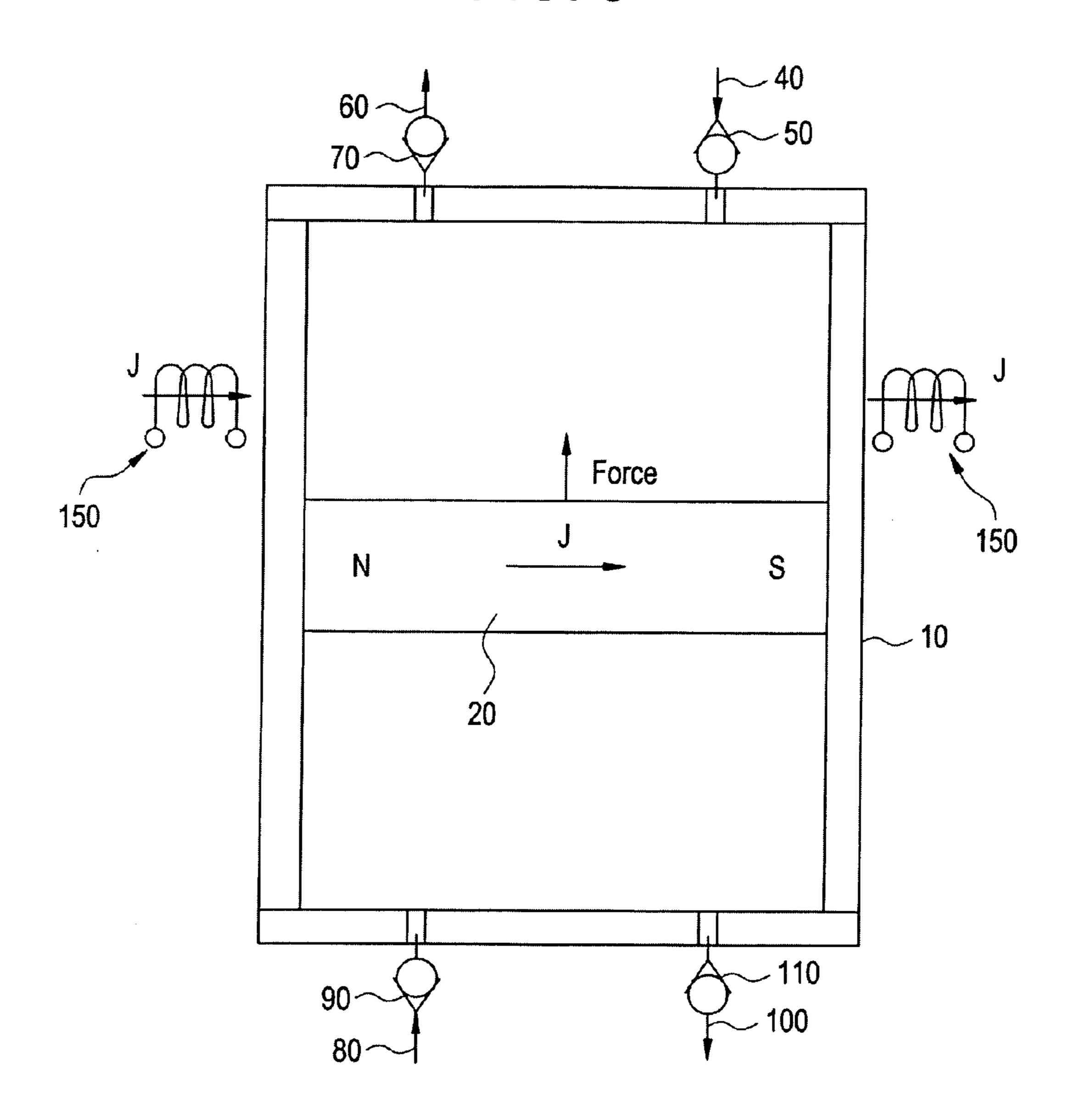


FIG. 4

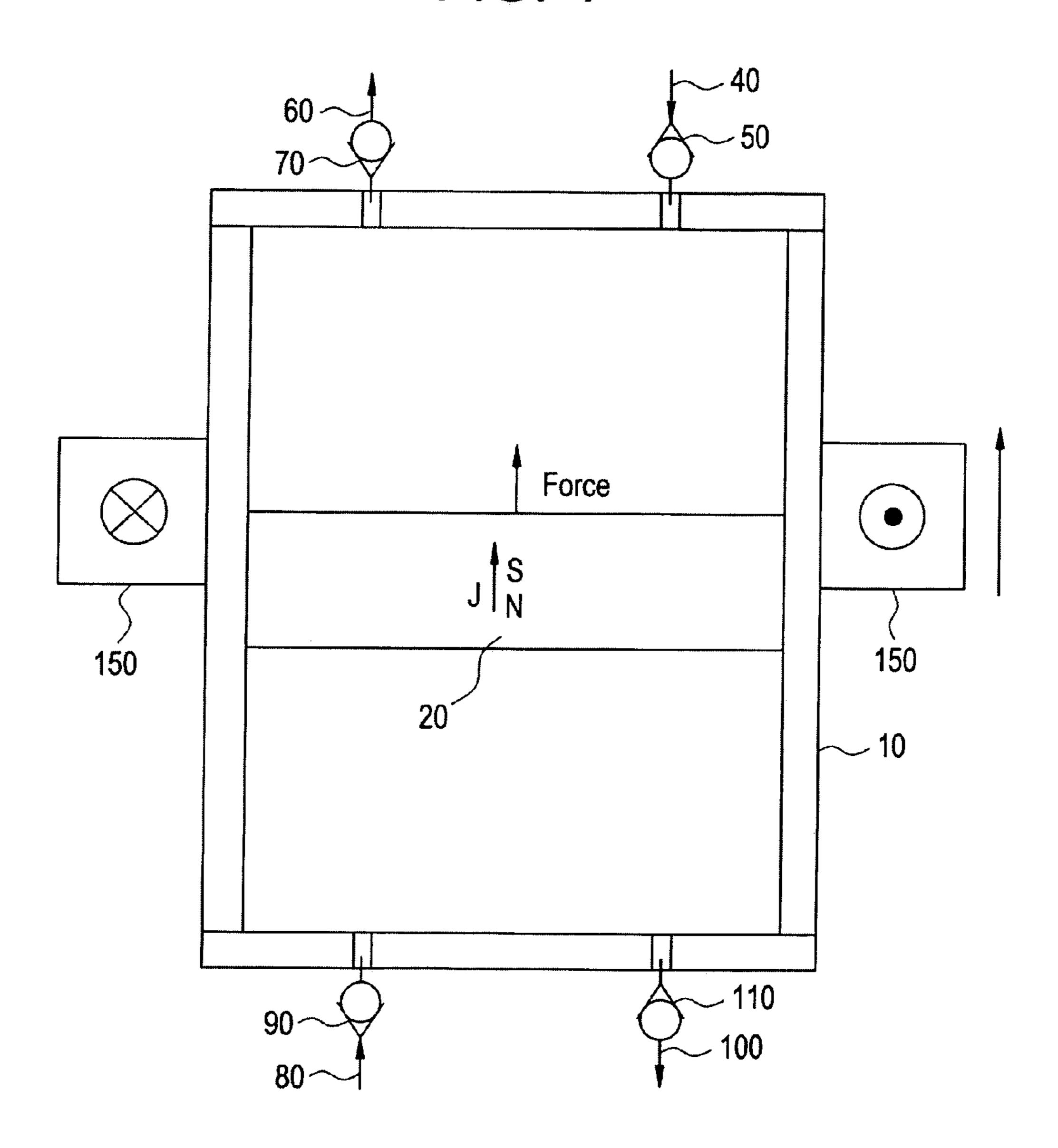


FIG. 5

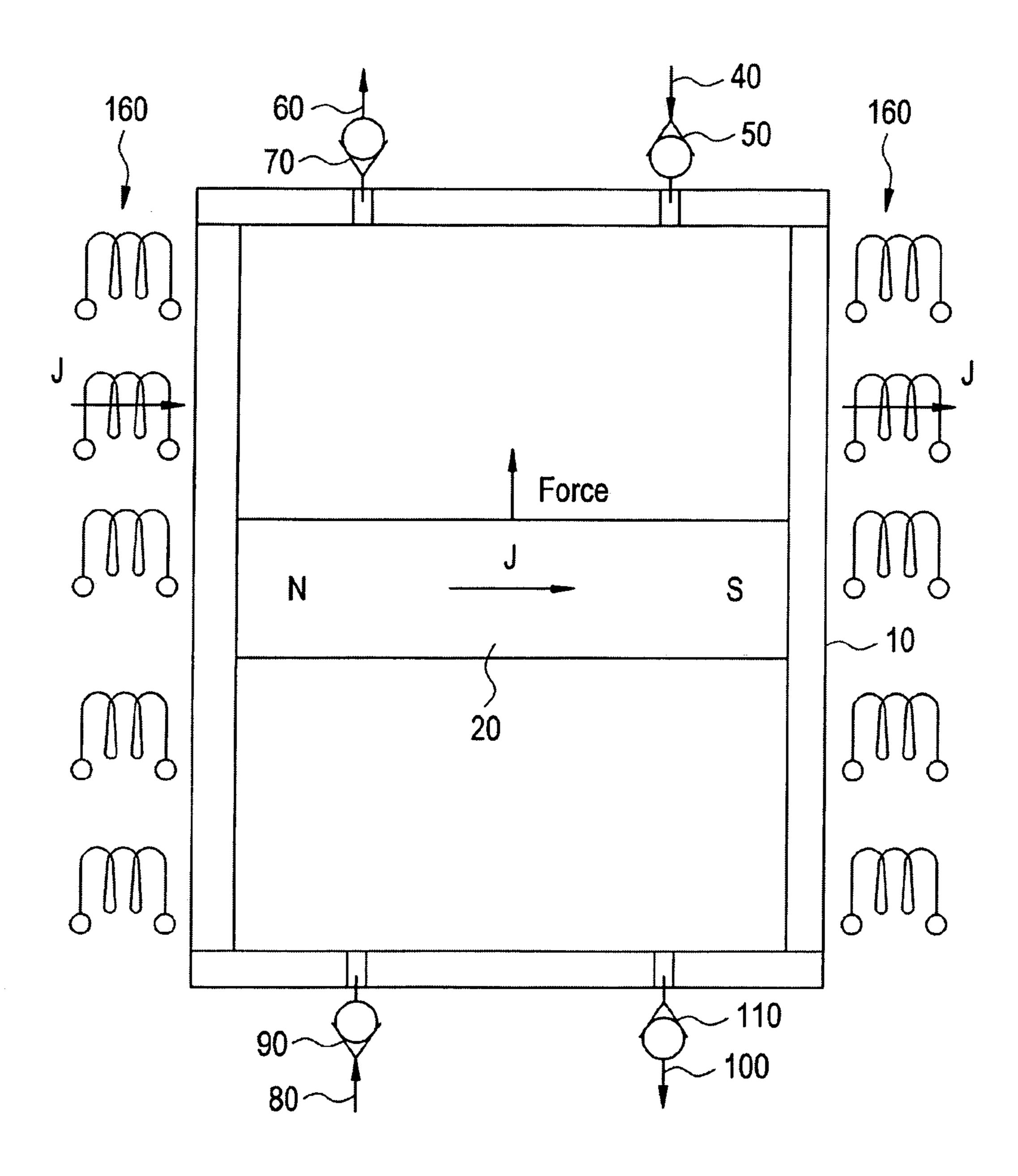
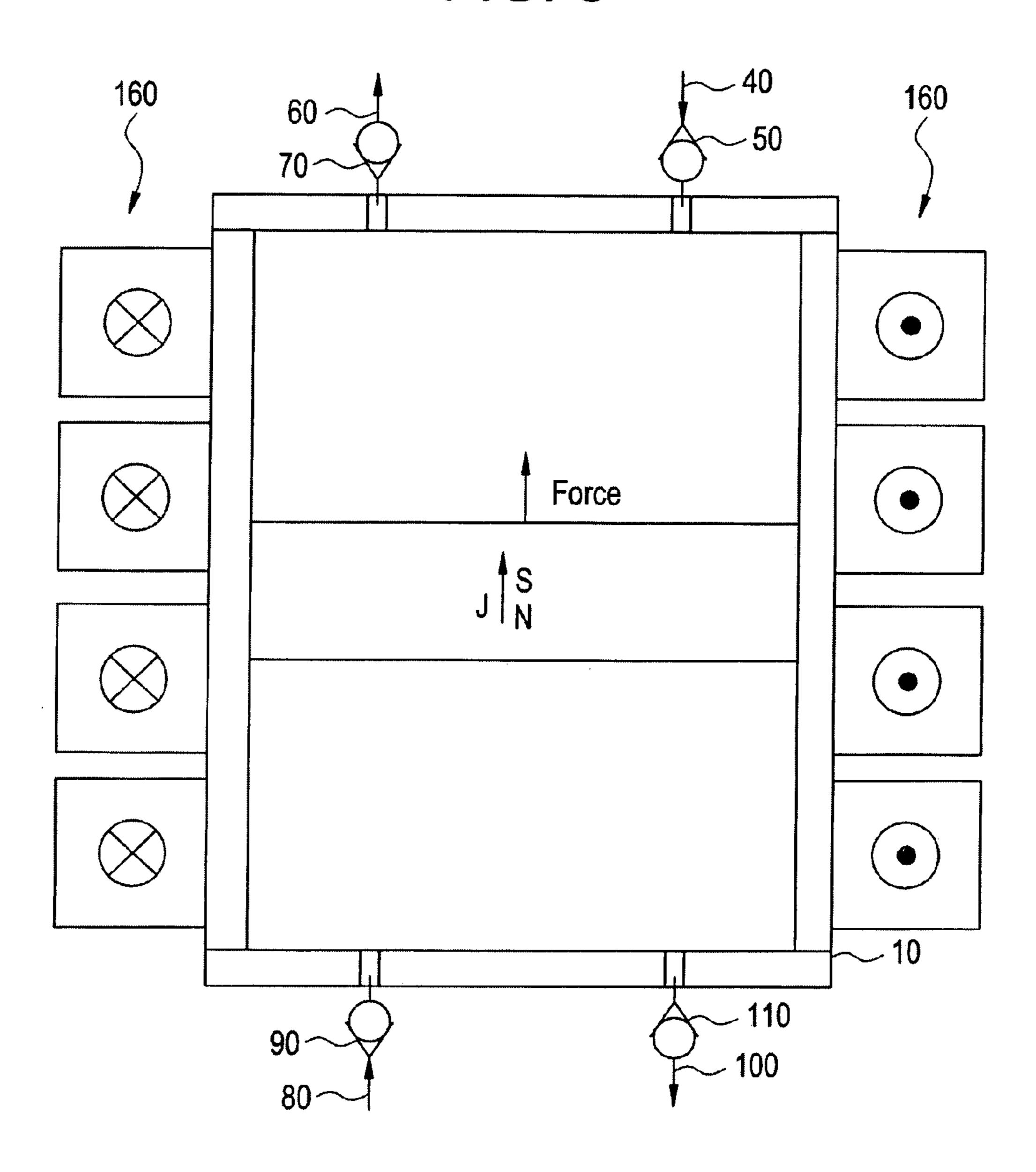
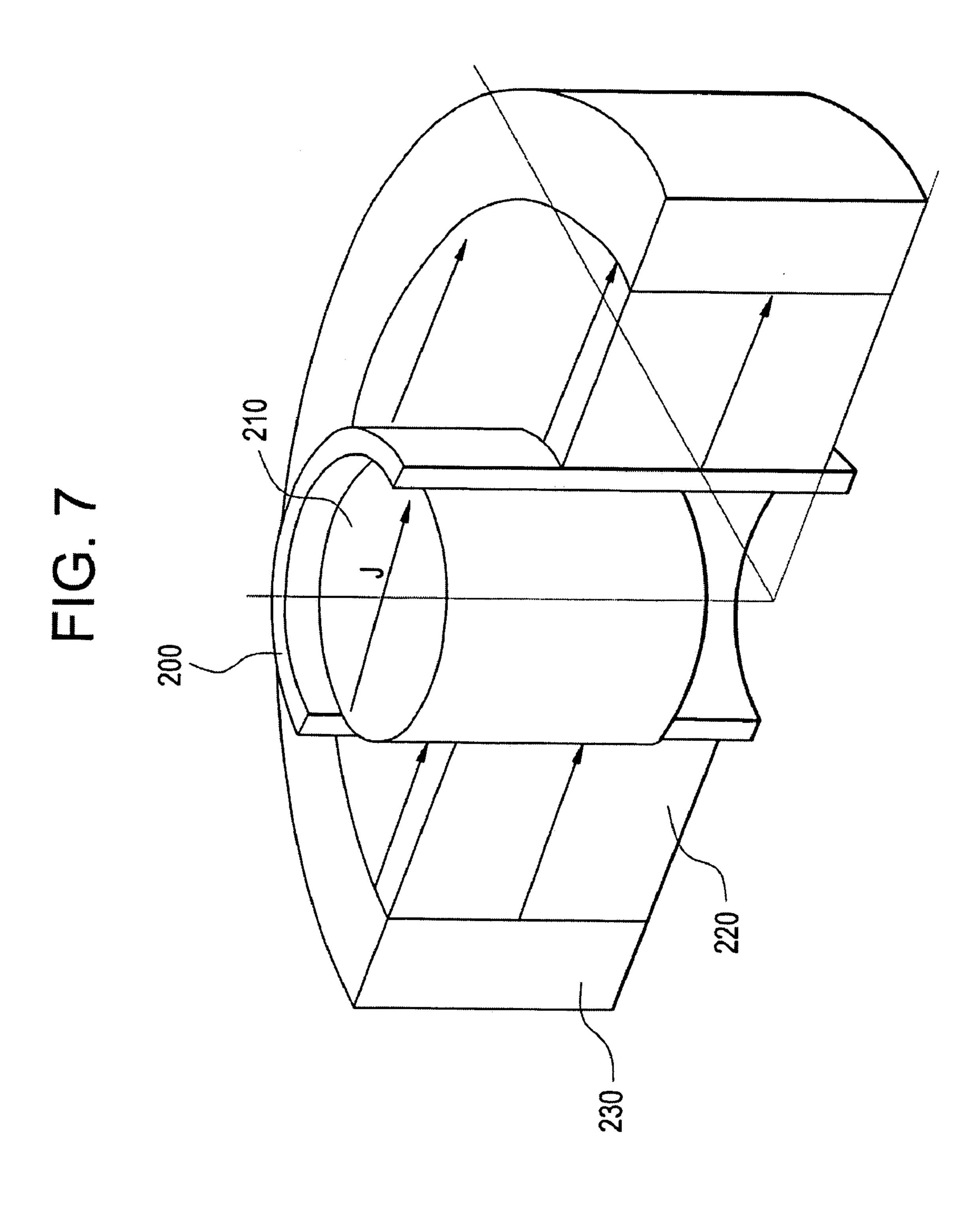
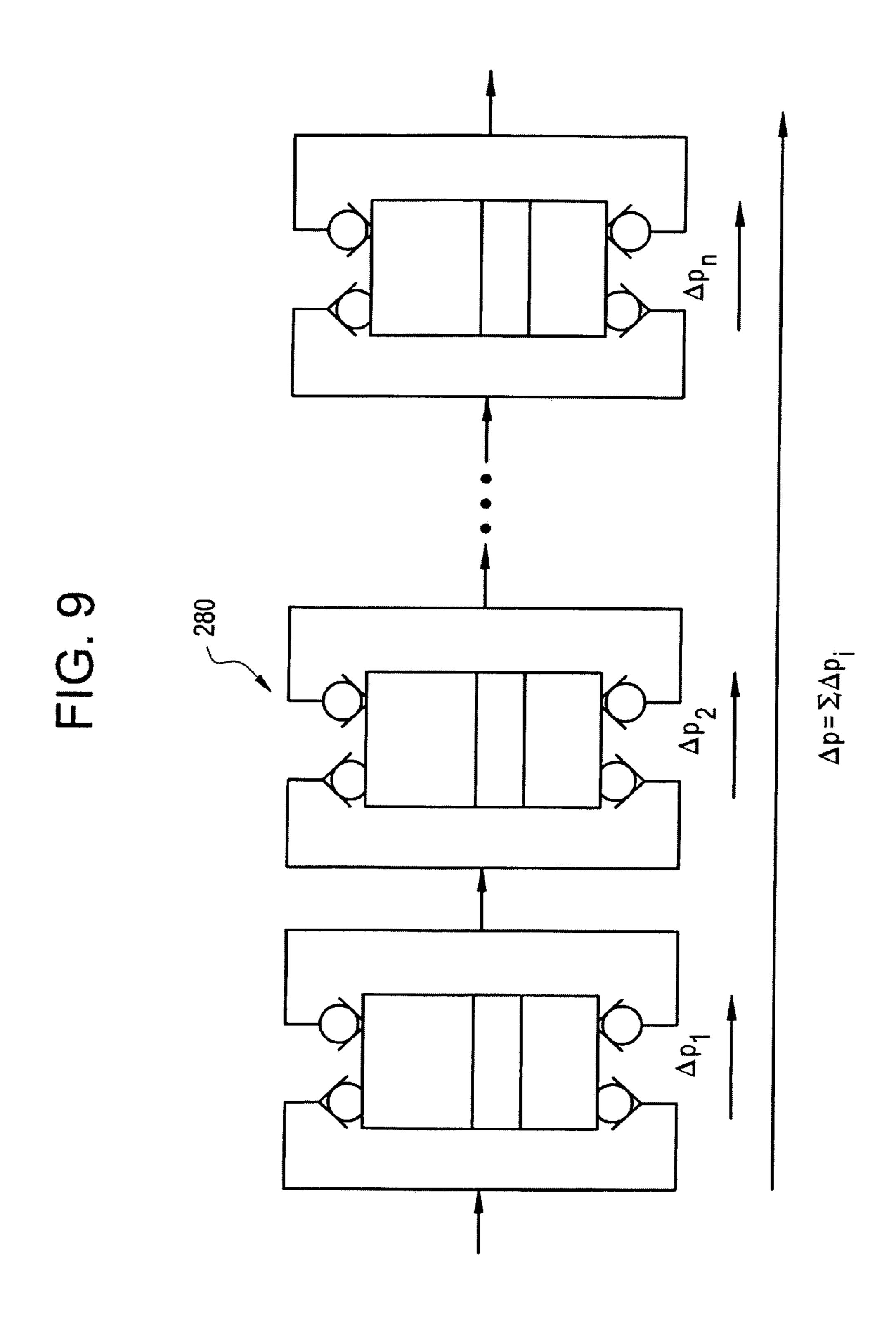


FIG. 6



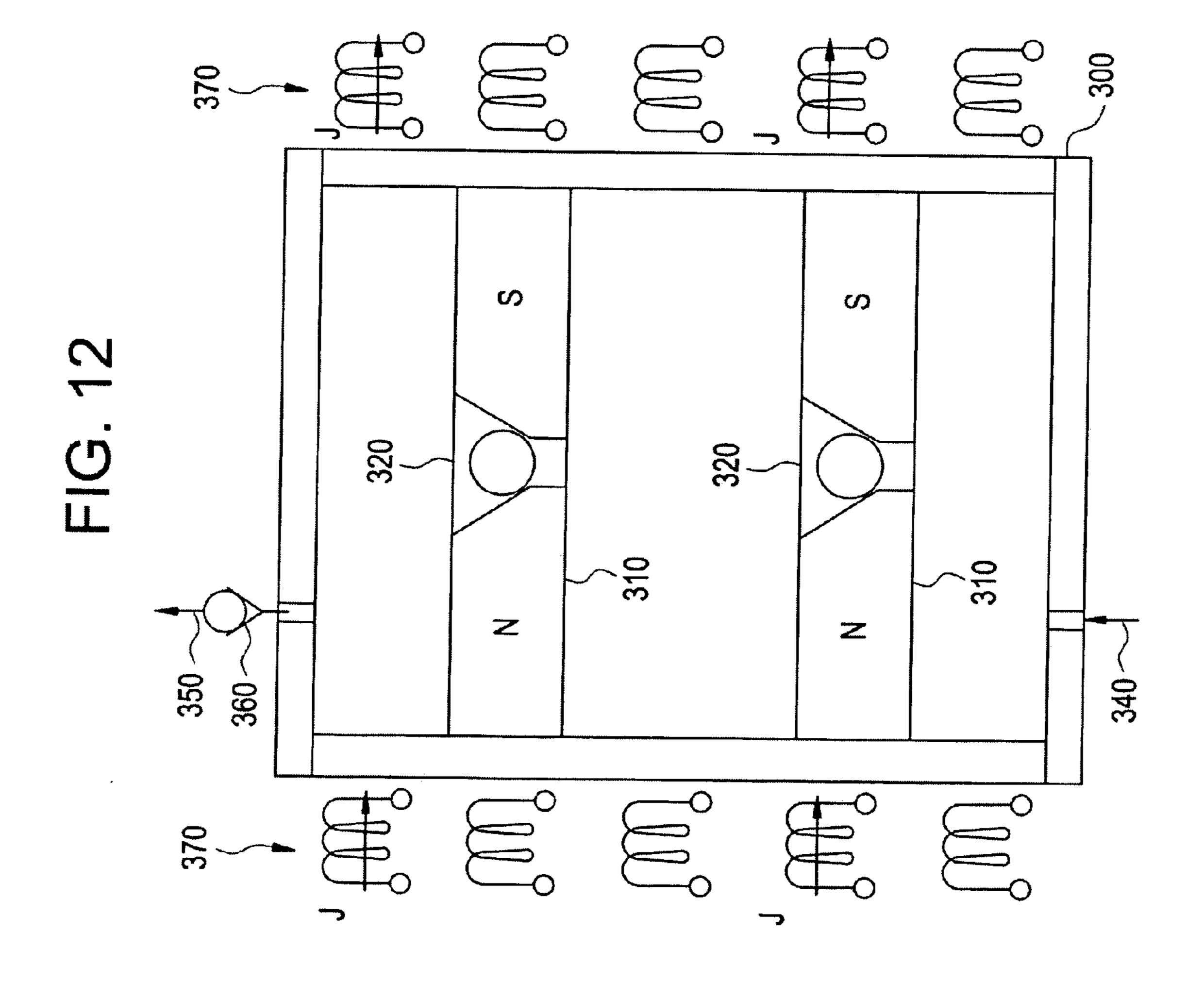


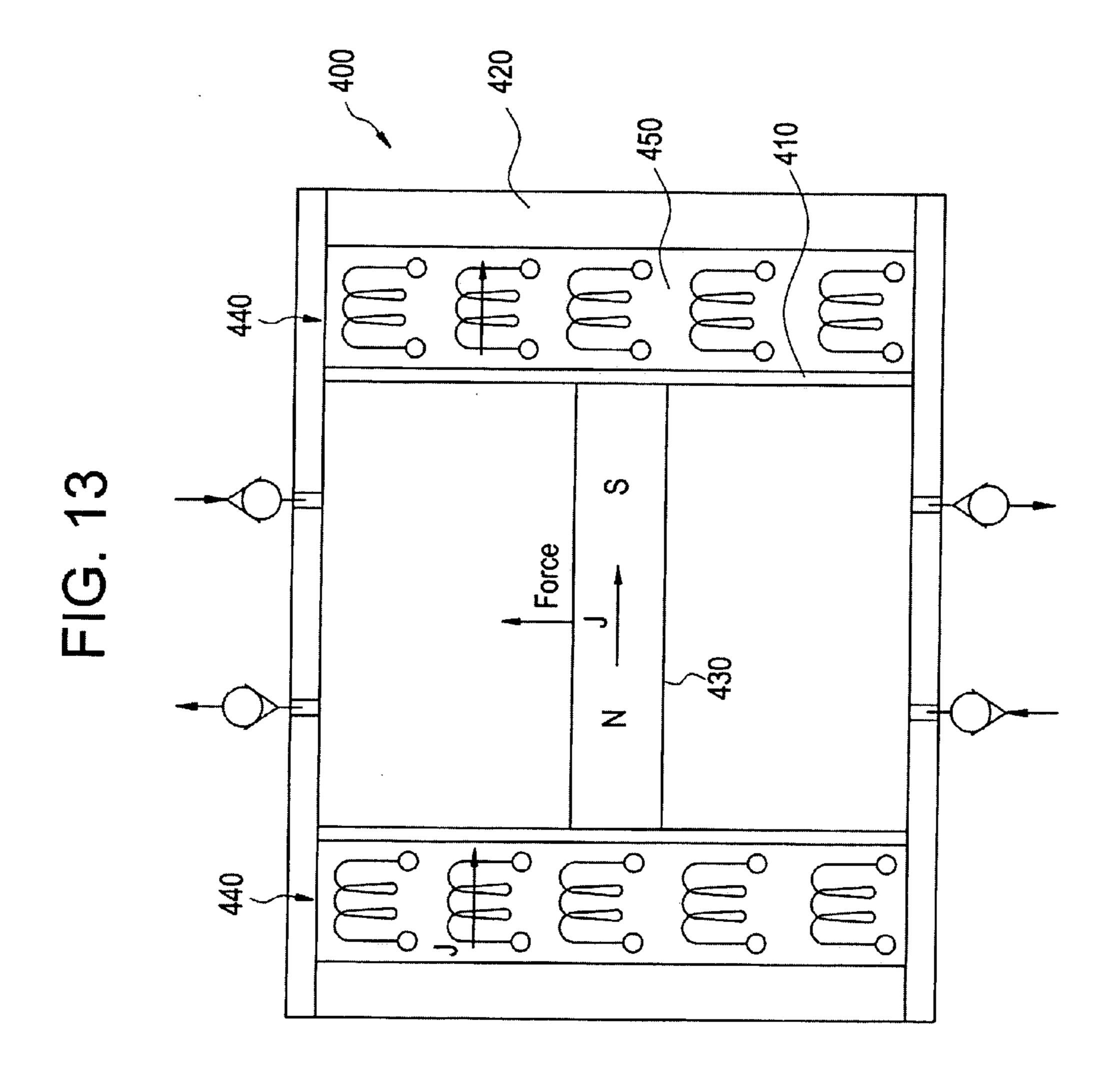
200



•E

S S Z 350 360 S





MAGNETIC PUMPING MACHINES

BACKGROUND

[0001] This disclosure relates generally to fluid movement and, more specifically, to magnetic pumping systems for displacing fluid.

[0002] Gas pumps or compressors are widely used for a variety of purposes. For example, they are used in pipeline transport to move natural gas from the production site to the consumer and in refrigeration and air conditioner equipment to move heat from one place to another. They are also used in industrial plants, such as petroleum refineries and chemical plants, to compress intermediate and end product gases. Further, they are used to deliver pressurized auxiliary gas (e.g., inert gas such as nitrogen or process gas such as natural gas) to the gas seals of a larger turbo compressor in order to maintain the sealing functionality of the seals.

[0003] Unfortunately, currently used gas pumps are susceptible to leaking, which can be very dangerous when the gas being compressed is highly corrosive or explosive. Gas pumps can be redesigned to fulfill the safety and reliability requirements of such challenging applications. However, this redesign can be costly and time consuming.

[0004] One type of gas pump is the reciprocating compressor, which employs a piston driven by a crankshaft to exert force on a fluid within a cylinder, thereby compressing it. Reciprocating compressors have several drawbacks. For example, they are usually powered by a near-by electric supply, which poses a safety hazard in those cases where combustible gases are being compressed. Also, reciprocating compressors have moving parts, e.g., the crankshaft, that compromise their reliability. Further, the flow output and the output pressure of reciprocating compressors are difficult to control.

[0005] A need therefore exists for improved pumps for conveying fluids such as gas.

SUMMARY

[0006] Disclosed herein are magnetic pumping systems. In one embodiment, a system for pumping fluid comprises: a substantially cylinder-shaped pump housing; a magnetic piston residing within the pump housing for displacing the fluid; a magnetic actuator surrounding the pump housing, wherein the magnetic actuator is configured to move the piston by a magnetic force between the magnetic piston and the magnetic actuator; and a ring comprising a magnetic element surrounding and attached to the magnetic actuator for increasing a magnitude of the magnetic force.

[0007] In an additional embodiment, a system for pumping fluid comprises: a pump housing; a magnetic piston residing within the pump housing for displacing the fluid; and an electromagnetic actuator outside the pump housing, wherein the electromagnetic actuator is configured to move the piston by an electromagnetic force.

[0008] In another embodiment, a system for pumping fluid comprises: a pump housing; a stack of at least two magnetic pistons residing within the pump housing for displacing the fluid; and a stack of at least two magnetic actuators outside the pump housing, wherein the magnetic actuators are configured to move the pistons by a magnetic force between the magnetic pistons and the magnetic actuators.

[0009] In yet another embodiment, a system for pumping fluid comprises: at least two magnetic pumps arranged in

series or in parallel, wherein each magnetic pump comprises: a pump housing; a magnetic piston residing within the pump housing for displacing the fluid; and an actuator outside the pump housing, wherein the actuator is configured to move the piston by a magnetic or electromagnetic force.

[0010] In still another embodiment, a system for pumping a fluid comprises: a pump housing comprising an inner wall laterally spaced from an outer wall; a magnetic piston residing within the inner wall for displacing the fluid; and a plurality of conductive coils arranged inside the outer wall along a periphery of the inner wall, wherein the piston is configured to move upon application of a current in a sequential pattern to the different conductive coils.

BRIEF DESCRIPTION OF THE DRAWINGS

[0011] Referring now to the Figures, which are exemplary embodiments, and wherein the like elements are numbered alike:

[0012] FIGS. 1-6 are cross-sectional views of magnetic pumping systems in accordance with various embodiments; [0013] FIG. 7 is a perspective view of a portion of a magnetic pumping system having two poles in accordance with an embodiment;

[0014] FIGS. 8a and 8b are perspective and cross-section views, respectively of a magnetic pumping system having multiple poles in accordance with an embodiment;

[0015] FIG. 9 is a flow diagram of magnetic pumps arranged in series;

[0016] FIG. 10 is a flow diagram of magnetic pumps arranged in parallel; and

[0017] FIGS. 11-13 are cross-sectional views of magnetic pumping systems in accordance with different embodiments.

DETAILED DESCRIPTION

Systems for pumping fluid that comprise magnetic [0018]pistons and permanent-magnetic or electromagnetic actuators for moving the pistons are described. In an embodiment, these pumping systems are suitable for use in high-pressure applications, e.g., greater than about 400 psi which, could be required in oil and gas production units. As used herein, the term "magnetic" refers to an ability to exhibit an attractive or repulsing force on other materials (i.e., magnetism) resulting from the quantum-mechanical spin and orbital motion of electrons, wherein magnetic objects/materials are considered to be "permanent" magnetic objects/materials since the magnetism occurs even when no electrical current is applied. As used herein, the term "electromagnetic" refers to an ability to exhibit an attractive force on other materials resulting from electric currents, wherein electromagnetic objects/materials are considered to be "temporary" magnetic objects/materials since a magnetic field is generated only when an electric current flows through the objects/materials. The pumping systems described herein are referred to as "magnetic pumping systems", meaning that they operate by means of a magnetic or electromagnetic force.

[0019] Tuning to FIG. 1, an embodiment of a magnetic pumping system is shown. The magnetic pumping system comprises a pump housing 10 for holding fluid, a magnetic piston 20 disposed inside the pump housing 10, and a permanent-magnetic actuator 30 disposed outside the pump housing 10. The pump housing 10 can be substantially cylinder-shaped, and the magnetic piston 20 can be sized to fit inside the pump housing 10. The magnetic piston 20 is capable of

sliding in a direction parallel to a central axis (not shown) of the pump housing 10. The magnetic actuator 30 can be in the form of a ring sized to fit around the pump housing 10. The piston 20 and the actuator 30 can comprise a permanent magnetic material. Examples of suitable permanent magnetic materials include but are not limited to metals such as steel, iron, iron ore (e.g., magnetite and lodestone), cobalt, zinc, nickel, and rare earth metals (e.g., gadolinium and dysprosium), alloys such as NdFeB, SmCo, and AlNiCo, composite materials such as AlNiCo in a plastic, and combinations comprising at least one of the foregoing materials. A suction line 40 and a discharge line 60 in fluid communication with the pump housing 10 are located on one end of the pumping system for allowing fluid to flow into and out of the pump housing 10. Check valves 50 and 70 are disposed in the suction line 40 and the discharge line 60, respectively. Another suction line 80 and another discharge line 100 having check valves 90 and 110, respectively, are located on the opposite end Of the pumping system from the suction and discharge lines 40 and 60.

[0020] The operation of the magnetic pumping system shown in FIG. 1 entails moving the magnetic actuator 30 by an external driver (not shown). The external driver can be, for example, an electric, hydraulic, pneumatic, or piezoelectric driver. The use of an electrical driver is less desired in cases where the fluid being displaced is explosive. Due to a magnetic force between the magnetic actuator 30 and the magnetic piston 20, the movement of the piston 20 can follow the movement of the actuator 30 such that it compresses the fluid in the forward chamber. The magnetic force between the piston 20 and the actuator 30 is the highest if the piston 20 is deflected at half of its length against the actuator 30. The magnetic polarization J of the piston 20 and the actuator 30 have the same orientation. In FIG. 1, J is oriented in the radial direction (i.e., parallel to the radius of the pump housing 10) with the south pole S and the north pole N of the magnetic field being oriented as shown.

[0021] The movement of the magnetic piston 20 can be controlled such that it repeatable slides forward in a first stroke that is parallel to an axis of the pump housing 10 and backward in a second stroke that is counter/opposite to the first stroke. When the piston 20 slides forward, the check valve 70 to the discharge line 60 can open as a result of the pressure inside the adjacent chamber rising above the pressure of the discharge line 60, allowing fluid to flow out of that chamber. At the same time, the pressure in the opposite chamber can drop, causing the check valve 90 to the suction line 80 to open such that fluid flows back into that chamber. When the piston 20 slides backward, the check valve 110 to the discharge line 100 can open to allow fluid to be forced out of the adjacent chamber. At the same time, the check valve 50 to the suction line 40 can open to allow fluid to flow back into the opposite chamber. Hence, both chambers can be used for fluid delivery.

[0022] Another embodiment of a magnetic pumping system is shown in FIG. 2. The magnetic pumping system of FIG. 2 is similar to the one depicted in FIG. 1 except that it contains multiple magnetic pistons 20 stacked together and multiple magnetic actuators 30 stacked together. Configuring the pistons 20 and the actuators 30 in this manner can achieve a greater magnetic force between the two. J of one of the magnetic pistons 20 and J of another of the magnetic pistons 20 are oriented in opposite directions.

[0023] FIGS. 3 and 4 illustrate additional embodiments of the magnetic pumping system with the same components as the pumping system depicted in FIG. 1. However, instead of a permanent-magnetic actuator, an electromagnetic actuator 150 is employed to move the magnetic piston 20 as a result of an electromagnetic force between the two. As shown, the electromagnetic actuator 150 can be a conductive coil 150 outside the pump housing 10 that is coupled to an electrical supply. The conductive coil 150 can comprise any conductive material such as a metal, e.g., copper or aluminum, or alloys of metals. The current in the conductive coil 150 can be switched on to operate the magnetic pumping system. Current flowing through the coil 150 can produce an electromagnetic field of sufficient strength to drag the magnetic piston 20 along as the coil 150 moves parallel to the axis of the pump housing 10. An external driver, e.g., an electric, hydraulic, pneumatic, or piezoelectric driver, can be used to cause the conductive coil to move in the desired direction.

[0024] The conductive coil 150 can be oriented as shown in FIG. 3 (i.e., with the axis of the coil in a radial direction) such that J is oriented in the radial direction. The coil 150 in FIG. 3 is wound around a permanent-magnetic core, e.g., a ferromagnetic core (not shown). The smaller the distance between the poles of the permanent-magnetic core and the magnetic piston, the higher the magnetic force between the two. Alternatively, the conductive coil 150 can be oriented as shown in FIG. 4 such that J is oriented in the axial direction (i.e., parallel to an axis of the pump housing 10). No permanent-magnetic core is needed in the design of FIG. 4.

[0025] Two magnetic pumping system designs/embodiments that overcome the problems associated with moving parts are depicted in FIGS. 5 and 6. In particular, moving parts are no longer present outside the pump housing 10. This elimination of moving parts can improve the robustness and reduce the costs (e.g., maintenance cost) of the magnetic pumping system. As shown in FIG. 5, several conductive coils 160 (also called stator coils) can be arranged at different fixed positions along the lateral periphery of the pump housing 10. The coils 160 can comprise any conductive material such as a metal, e.g., copper or aluminum, or alloys of different metals. The coils 160 can be coupled to an electric supply. By switching the current through the coils 160 on and off in a defined sequential pattern, a moving electromagnetic field can be produced that attracts and thus moves the magnetic piston 20. Thus, the coils 160 do not have to be moved by an external driver to place the piston 20 in motion. Accordingly, the pumping system is absent of moving parts outside of the pump housing 10.

[0026] The conductive coils 160 can be oriented as shown in FIG. 5 to allow J to be oriented in a radial direction when current is flowing through the coils 160. The coils 160 in FIG. 5 are wound around a permanent-magnetic core, e.g., a ferromagnetic core (not shown). The smaller the distance between the poles of the permanent-magnetic core and the magnetic piston, the higher the magnetic force between the two. Alternatively, the coils 160 can be oriented as shown in FIG. 6 to allow J to be oriented in an axial direction when current is flowing through the coils 160. No permanent-magnetic core is needed in the design of FIG. 6.

[0027] Turning to FIG. 7, an embodiment of a magnetic pumping system having a two-pole design is depicted. The pump housing 200 can be substantially cylinder-shaped. The magnetic piston 210 can be a single cylinder-shaped permanent magnet, and the magnetic actuator 220 can be a single

permanent magnet that surrounds the pump housing 200. Also, an additional permanent-magnetic ring 230 made of a ferromagnetic material, e.g., iron or steel, can surround and be attached to the magnetic actuator 220. The aforementioned examples of magnetic materials are appropriate for use in the piston 210 and the actuator 220. The magnetic pumping system shown in FIG. 7 operates in the same way as the previously described pumping systems with the exception that the additional magnetic ring 230 and the magnetic actuator 220 can be moved together using an external driver (e.g., a pneumatic driver, a hydraulic driver, a piezoelectric driver, or an electric driver) such that two magnetic fields are produced. The magnetic polarization J of each field is oriented in the same radial direction. The ring 230 encloses the actuator 220 to provide a return path for the magnetic flux, thus increasing the magnitude of the magnetic force between the piston 200 and the actuator 2200/ring 230 assembly. This increased magnetic force and the compact design of this pumping system provide for excellent performance.

[0028] A multi-pole design of a magnetic pumping system in accordance with an embodiment is shown in FIGS. 8a and 8b. The magnetic piston 210 can include a permanent-magnetic core 240 made of a ferromagnetic material, e.g., iron or steel, and multiple permanent magnets 250 arranged around a periphery of the core 240. The core 240 is shown as being hexagonal in shape with six magnets 250 encircling the core **240**. It is understood that the core **240** can take the form of other shapes as long as the magnets 250 can be positioned snug against the outside of the core 240. The magnetic actuator 220 also can include multiple permanent magnets 260 aligned to the piston magnets 250. A permanent-magnetic ring 270 made of a ferromagnetic material, e.g. iron or steel, can surround and be attached to the actuator magnets 260. This magnetic pumping system operates in a manner similar to the pumping system shown in FIG. 7. However, six magnetic fields rather than two can be produced to aid the movement of the piston 210. The magnetic flux thus can be guided from any actuator magnet 260 through the pump housing 200 to the opposite piston magnet 250 via the permanent-magnetic core 240 to the next piston magnet 250, back through the pump housing 200 again, past the opposite actuator magnet 260 and the permanent-magnetic ring 270, and back to the same actuator magnet 260 where it started. The magnetic polarization J of each piston magnet 250 and each actuator magnet **260** is depicted in FIG. **8***b*.

[0029] FIGS. 9 and 10 illustrate that a series of magnetic pumps can be cascaded in sequential or parallel order, respectively. The pumping system 280 shown in FIG. 9 includes magnetic pumps 280 arranged in series to increase the differential pressure ΔP across the entire pumping system. The pumping system shown in FIG. 10 includes magnetic pumps 290 arranged in parallel to increase the mass flow rate, m, through the entire pumping system. Any of the previously described pumping systems or any combinations thereof can be cascaded as shown in FIGS. 9 and 10.

[0030] Yet another embodiment of a magnetic pumping system is shown in FIG. 11. This system can include a pump housing 300 for holding the fluid being compressed, a magnetic piston 310 for displacing the fluid, and a magnetic actuator 330 that can cause the motion of the magnetic piston 310 via a magnetic force between the piston 310 and the actuator 330. An external driver (not shown) such as a hydraulic, pneumatic, piezoelectric, or electric driver can be utilized to move the actuator 330. The magnetic components can

comprise any of the previously mentioned magnetic materials. A suction line 340 and a discharge line 350 in fluid communication with the pump housing 300 can be located at opposite ends of the pump system. Due to a lack of space on the pump housing 300, a suction check valve 320 can be located on piston 310 instead of in the suction line 340. The check valve 320 can open and close due to the differential pressure along the valve 320 without any additional actuation. Hence, more space is available on the pump housing 300 for a discharge valve 360. Therefore, the flow cross-section of the suction and discharge valves 320 and 360 can be increased to reduce valve losses. The previously described embodiments could also incorporate a check valve in the magnetic piston to provide for more space along the pump housing.

[0031] In an additional embodiment, a pumping system can include two or more magnetic pistons 310 cascaded in a spaced apart orientation as shown in FIG. 12. Optionally, the pistons 310 can include check valves 320 to permit the flow of fluid through the pistons 310. Multiple conductive coils 370 coupled to an electric supply can be arranged at the periphery of the pump housing 300. Each of the conductive coils 370 can be wound around a permanent-magnetic core comprising a ferromagnetic material such as iron (not shown). An electric current can be applied to two of the coils 370 at a time in a defined sequential pattern to move both of the magnetic pistons 310. A specific phase shift can be applied between the movements of the pistons 310 to boost differential pressure and to reduce pressure pulsations. The distance between the poles of the magnetic pistons 310 and the poles of the coils 370 (i.e., the actuator poles) determines the magnetic force there between and thus dictates the maximum differential pressure delivered by the pumping system. However, the amount by which this distance can be reduced is limited because the wall thickness of the pump housing 300 needs to be sufficient to withstand a specific amount of pressure.

[0032] A solution to this limitation on the reduction of the distance between the magnetic piston and the conductive coils is illustrated by the magnetic pumping system shown in FIG. 13. The magnetic pumping system can include a pump housing 400 comprising an inner wall 410 laterally spaced from an outer wall 420, wherein the outer wall 420 is thicker than the inner wall 410. A magnetic piston 430 can be disposed within the inner wall 410 for displacing fluid within the pump housing 400. Further, a plurality of conductive coils 440 can be arranged inside the outer wall 420 along a periphery of the inner wall 410 (i.e., between the outer wall 420 and the inner wall 410). Each of the conductive coils 440 can be wound around a permanent-magnetic core comprising a ferromagnetic material such as iron (not shown). As in previous embodiments, a current can be applied in a defined sequential pattern to the different coils 440 to cause the piston 430 to move. The relatively thin inner wall 410 can separate the fluid from the coils 440, and the thicker outer wall 420 can serve to withstand the pressure forces. The intermediate space unoccupied by the coils 440 can be filled with a suitable solid or liquid, such as mineral oil, to transfer the pressure forces from the inner wall **410** to the outer wall **420**. Examples of suitable filler materials include, but are not limited to, epoxy, mineral oil, a thermoplastic, and combinations comprising at least one of the foregoing filler materials. Additionally, this filling improves heat removal from the coils 440 and thus enables the increase of power density.

[0033] The magnetic pumping systems described above have several advantages over currently used pumping sys-

tems. For example, the magnetic actuator and the magnetic piston are not physically connected, thus allowing the pump housing to be "hermetically sealed". As used herein, "hermetically sealed" refers to the ability of the pump housing to prevent any material from moving into or out of the pump housing except through a line or conduit connected to the pump housing. As such, the fluid passing through the pump housing cannot escape into the atmosphere. This benefit is especially useful when the fluid being delivered is corrosive or explosive. Further, the pumping systems can be actuated without utilizing a high power electrical supply that could pose a danger if placed near a combustible and explosive fluid. In addition, the pumping systems can be volumetric acting devices that deliver the fluid in an unsteady pulsating mode. Thus, in contrast to other volumetric acting machines, the flow output and the output pressure can be controlled easily by a suitable actuating mechanism, e.g., by the aforementioned electrically driven actuator. Another advantage is that moving parts outside the pump housing can be avoided by employing conductive coils to actuate the magnetic piston, thereby improving reliability and reducing cost. Keeping the actuator and the piston separated also makes the system modular, meaning that several pumps can be easily stacked together in series or in parallel to increase differential pressure and/or mass flow.

[0034] As used herein, the terms "a" and "an" do not denote a limitation of quantity, but rather denote the presence of at least one of the referenced items. Reference throughout the specification to "one embodiment", "another embodiment", "an embodiment", and so forth means that a particular element (e.g., feature, structure, and/or characteristic) described in connection with the embodiment is included in at least one embodiment described herein, and may or may not be present in other embodiments. In addition, it is to be understood that the described elements may be combined in any suitable manner in the various embodiments. Unless defined otherwise, technical and scientific terms used herein have the same meaning as is commonly understood by one of skill in the art to which this invention belongs.

[0035] While the invention has been described with reference to exemplary embodiments, it will be understood by those skilled in the art that various changes may be made and equivalents may be substituted for elements thereof without departing from the scope of the invention. In addition, many modifications may be made to adapt a particular situation or material to the teachings of the invention without departing from the essential scope thereof. Therefore, it is intended that the invention not be limited to the particular embodiment disclosed as the best mode contemplated for carrying out this invention, but that the invention will include all embodiments falling within the scope of the appended claims.

What is claimed is:

- 1. A system for pumping fluid, comprising:
- a substantially cylinder-shaped pump housing;
- a magnetic piston residing within the pump housing for displacing the fluid;
- a magnetic actuator surrounding the pump housing, wherein the magnetic actuator is configured to move the piston by a magnetic force between the magnetic piston and the magnetic actuator; and
- a magnetic ring surrounding and attached to the magnetic actuator for increasing a magnitude of the magnetic force.

- 2. The system of claim 1, further comprising an external driver for moving the magnetic actuator and the magnetic ring, wherein the external driver is capable of moving the magnetic actuator and the ring repeatedly in a first direction parallel to an axis of the pump housing followed by a second direction opposite the first direction.
- 3. The system of claim 2, wherein the external driver is selected from the group consisting of a pneumatic driver, a hydraulic driver, a piezoelectric driver, and an electric driver.
- 4. The system of claim 1, wherein the fluid is hermetically sealed within the pump housing when the fluid passes through the pump housing.
- 5. The system of claim 1, wherein the magnetic piston is substantially cylinder-shaped, and wherein the magnetic ring comprises iron.
- 6. The system of claim 1, wherein the magnetic piston comprises a magnetic core surrounded by piston magnets arranged around a periphery of the magnetic core, and wherein the magnetic actuator comprises a plurality of actuator magnets aligned to the piston magnets.
- 7. The system of claim 1, comprising two or more pumps arranged in series or in parallel, wherein each pump comprises the pump housing, the magnetic piston, the magnetic actuator, and the ring.
 - 8. A system for pumping fluid, comprising:
 - a pump housing;
 - a stack of at least two magnetic piston residing within the pump housing for displacing the fluid; and
 - a stack of at least two magnetic actuators outside the pump housing, wherein the magnetic actuators are configured to move the pistons by a magnetic force between the magnetic pistons and the magnetic actuators.
- 9. The system of claim 8, further comprising an external driver for moving the at least two magnetic actuators, wherein the external driver is configured to move the at least two magnetic actuators repeatedly in a first direction parallel to an axis of the pump housing followed by a second direction opposite the first direction.
- 10. The system of claim 8, wherein the fluid is hermetically sealed within the pump housing when the fluid passes through the pump housing.
 - 11. A system for pumping fluid, comprising:
 - at least two magnetic pumps arranged in series or in parallel, wherein each magnetic pump comprises:
 - a pump housing;
 - a magnetic piston residing within the pump housing for displacing the fluid; and
 - an actuator outside the pump housing, wherein the actuator is configured to move the piston by a magnetic or electromagnetic force.
 - 12. A system for pumping a fluid, comprising:
 - a pump housing comprising an inner wall laterally spaced from an outer wall,
 - a magnetic piston residing within the inner wall for displacing the fluid; and
 - a plurality of conductive coils arranged inside the outer wall along a periphery of the inner wall, wherein the piston is configured to move upon application of a current in a sequential pattern to the different conductive coils.
- 13. The system of claim 12, wherein the outer wall is thicker than the inner wall.

- 14. The system of claim 12, wherein a solid or liquid material fills unoccupied space provided between the inner wall and the outer wall.
- 15. The system of claim 12, wherein the fluid is hermetically sealed within the pump housing when the fluid passes through the pump housing.
 - 16. A system for pumping fluid, comprising: a pump housing;
 - a magnetic piston residing within the pump housing for displacing the fluid;
 - an electromagnetic actuator outside the pump housing comprising a movable conductive coil, wherein the electromagnetic actuator is configured to move the piston by an electromagnetic force; and
 - an external driver for moving the conductive coil.

- 17. The system of claim 16, wherein the external driver is configured to move the conductive coil repeatedly in a first direction parallel to an axis of the pump housing followed by a second direction opposite the first direction.
- 18. The system of claim 16, wherein the fluid is hermetically sealed within the pump housing when the fluid passes through the pump housing.
- 19. The system of claim 16, further comprising one or more additional magnetic pistons spaced from and in parallel to the magnetic piston within the pump housing.
- 20. The system of claim 16, comprising two or more pumps arranged in series or in parallel, wherein each pump comprises the pump housing, the magnetic piston, and the electromagnetic actuator.

* * * *