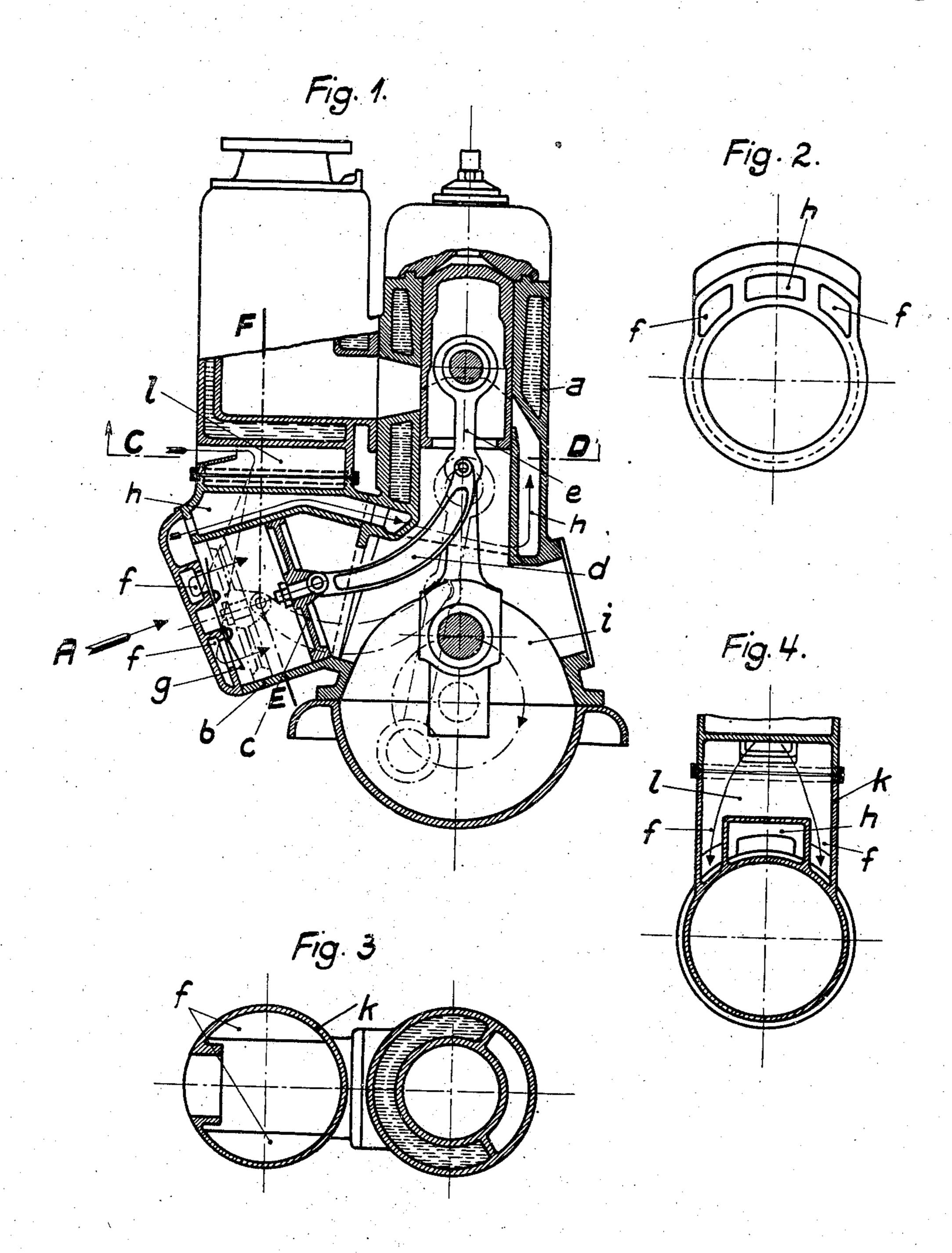
Oct. 7, 1930.

TWO-STROKE CYCLE INTERNAL COMBUSTION ENGINE EQUIPPED WITH SCAVENGING PUMP ANNEX Filed Dec. 12, 1928

2 Sheets-Sheet 1



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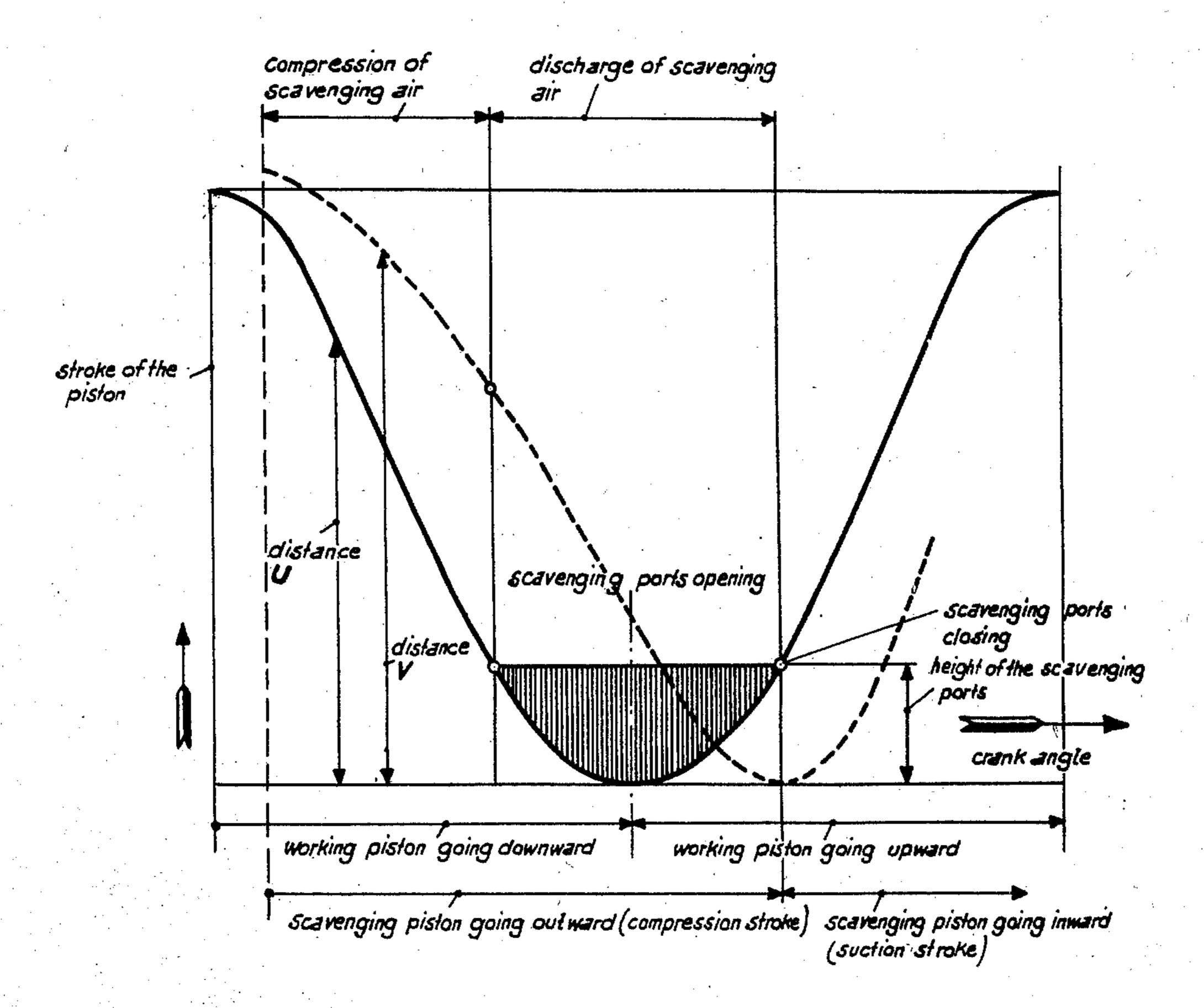
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R. GAUPP LE INTERNAL COMBUSTION ENGINE

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Fig. 5.



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PUMP ANNEX

Application filed December 12, 1928, Serial No. 325,517, and in Germany December 17, 1927.

My invention, which relates to internal ing cylinder and crank shaft as above dean embodiment of the invention.

10 an engine constructed according to the in- necting link toward the crank shaft. Fur- 60

vention.

In the drawings:—

engine with parts in elevation;

Fig. 2 is a view of the pump cylinder, with the cylinder head removed, looking at the cylinder in the direction of the arrow A;

Fig. 1;

Fig. 1; and

Fig. 5 is a diagram showing the relation between the movements of the air pump piston and working piston and the events and

25 periods of the cycles of each.

Referring to the drawing, a is the cylinder block of a vertical two stroke cycle Diesel engine. Attached to said cylinder block is the cylinder block b of the scavenging pump, the pump piston has finished part of its in-30 the piston c of the latter being joined with ward stroke, thus scavenging air under pres- so 35 through the channel h to the scavenging ports compression stroke, just as the scavenging 85 k, placed between exhaust pot and the air ports are open. The arrangement of the air conduits asso- invention herein described without departing ciated with the pump in the present embodi-from the spirit of the invention. ment of the invention is shown in Figs. 2 to 4. What I claim is: inder block to above that required to accom-

It will be observed that by arranging the scavenging pump with relation to the work- engine crank shaft having an engine crank, 100

modate the pump piston.

combustion engines, and has among its ob-scribed, and driving it in the manner above jects the driving and positioning of the described, the height of the engine measured scavenging pump and associated parts in an from the axis of the crank shaft to the top 5 improved manner, will be best understood of the working cylinder is reduced to a mini- 55 from the following description when read in mum, this result being accentuated by conthe light of the accompanying drawings of necting the scavenging pump piston to the connecting rod between the ends of the latter The drawings illustrate one embodiment of and further accentuated by bending the conther, the arrangement permits the placing of the expansion chamber l between the exhaust Fig. 1 is a longitudinal axial section of the pot and pump cylinder block b, which provides compactness and ample space for the expansion chamber sufficient to permit effec- 65 tive muffling.

The displacement of the scavenging and Fig. 3 is a section on the line C—D of working pistons for different positions of the crank shaft is shown diagrammatically in Fig. 4 is a section on the line F—E of Fig. 5. In this figure the abscissas denote 70 the different angles of the crank shaft, while the ordinates denote the volume displaced by the working and scavenging pistons, the full line curve representing the relations for the working piston and the dotted line curve the 75

relations for the scavenging piston.

Referring to the diagram, it will be noted that the scavenging ports begin to open when the main-connecting rod e by means of the sure is immediately available when the connecting rod d. The scavenging air is scavenging ports open. Further, it will be drawn in through the partially visible chan- noted that the pump piston reaches its inner nels f and the suction valves g and forced dead center, that is to say, the end of its of the working cylinder. The crank case i ports close or begin to close, and consequently has no connection with the scavenging ports. scavenging air is discharged into the working The suction chamber l is formed by the wall cylinder during all the time the scavenging

40 pump cylinder. The arrows indicate the It will be understood that wide deviations co courses of the inflowing and discharged air. may be made from the embodiment of the

45 It will be observed that this arrangement 1. A single acting non-reversible internal co does not increase the width of the pump cyl- combustion engine having, in combination, an engine cylinder having an exhaust port and a scavenging port in the wall thereof, an engine piston for controlling said ports, an

a connecting rod connecting said engine pis- an engine cylinder having an exhaust port engine cylinder which is occupied by said a connecting rod connecting said engine pisconnecting rod when said engine piston ton and engine crank, a scavenging pump pump being single acting with its working engine cylinder which is occupied by said chamber at the side of said pump piston remote from said engine cylinder, a link bent makes its compression stroke and being at toward the crank shaft connecting said con- an obtuse angle to said engine cylinder, the necting rod at an intermediate portion of its axial line of said pump cylinder extending length to said pump piston for reciprocating through the space between said crank shaft 15 the latter, and ports and passages connect- and said piston, and a link bent toward the 80 ing the working chamber of said pump to crank shaft connecting said connecting rod said scavenging port for delivering fresh air at a point materially removed from each end thereto.

2. A single acting non-reversible internal ing the latter. 20 combustion engine having, in combination, 5. A single acting non-reversible internal 85 25 crank, a connecting rod connecting said en- engine crank shaft having an engine crank, 90 of said engine cylinder which is occupied by said pump cylinder being at that side of 30 said connecting rod when said engine piston said engine cylinder which is occupied by 95 makes its compression stroke and being at said connecting rod when said engine piston an obtuse angle to said engine cylinder, said makes its compression stroke and being at pump being single acting with its working an obtuse angle to said engine cylinder, said chamber at the side of said pump piston re- pump being single acting with its working 35 mote from said engine cylinder, a link con- chamber at the side of said pump piston re- 100 necting said connecting rod at an interme- mote from said engine cylinder, and a link diate portion of its length to said pump bent toward said crank shaft connecting said piston for reciprocating the latter, an exhaust pot at the side of the engine cylinder its length to said pump piston for reciprocat-40 above said scavenging pump, and an expan- ing the latter. sion chamber for the intake of said pump between said pot and pump.

3. A single acting non-reversible internal combustion engine having, in combination, 45 an engine cylinder having an exhaust port and a scavenging port in the wall thereof, an engine piston for controlling said ports, an engine crank shaft having an engine crank, a connecting rod connecting said en-50 gine piston and engine crank, a scavenging pump having a pump cylinder and pump piston, said pump cylinder being at that side of said engine cylinder which is occupied by said connecting rod when said enb gine piston makes its compression stroke and being at an obtuse angle to said engine cylinder, the axial line of said pump cylinder extending through the space between said crank shaft and said piston, and a link connecting said connecting rod at a point materially removed from each end thereof to said pump piston for reciprocating the latter.

4. A single acting non-reversible internal 65 combustion engine having, in combination,

ton and engine crank, a scavenging pump and a scavenging port in the wall thereof, an having a pump cylinder and pump piston, engine piston for controlling said ports, an said pump cylinder being at that side of said engine crank shaft having an engine crank, makes its compression stroke and being at having a pump cylinder and pump piston, an obtuse angle to said engine cylinder, said said pump cylinder being at that side of said thereof to said pump piston for reciprocat-

an engine cylinder having an exhaust port combustion engine having, in combination, and a scavenging port in the wall thereof, an engine cylinder having an exhaust port an engine piston for controlling said ports, and a scavenging port in the wall thereof, an an engine crank shaft having an engine engine piston for controlling said ports, an gine piston and engine crank, a scavenging a connecting rod connecting said engine pispump having a pump cylinder and pump pis- ton and engine crank, a scavenging pump ton, said pump cylinder being at that side having a pump cylinder and pump piston, connecting rod at an intermediate portion of

In testimony whereof, I have signed my name to this specification.

RUDOLF GAUPP.

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