P. SCHRÖDER

CONDENSER FOR AIRSHIPS

Filed June 4, 1926

3 Sheets-Sheet 1

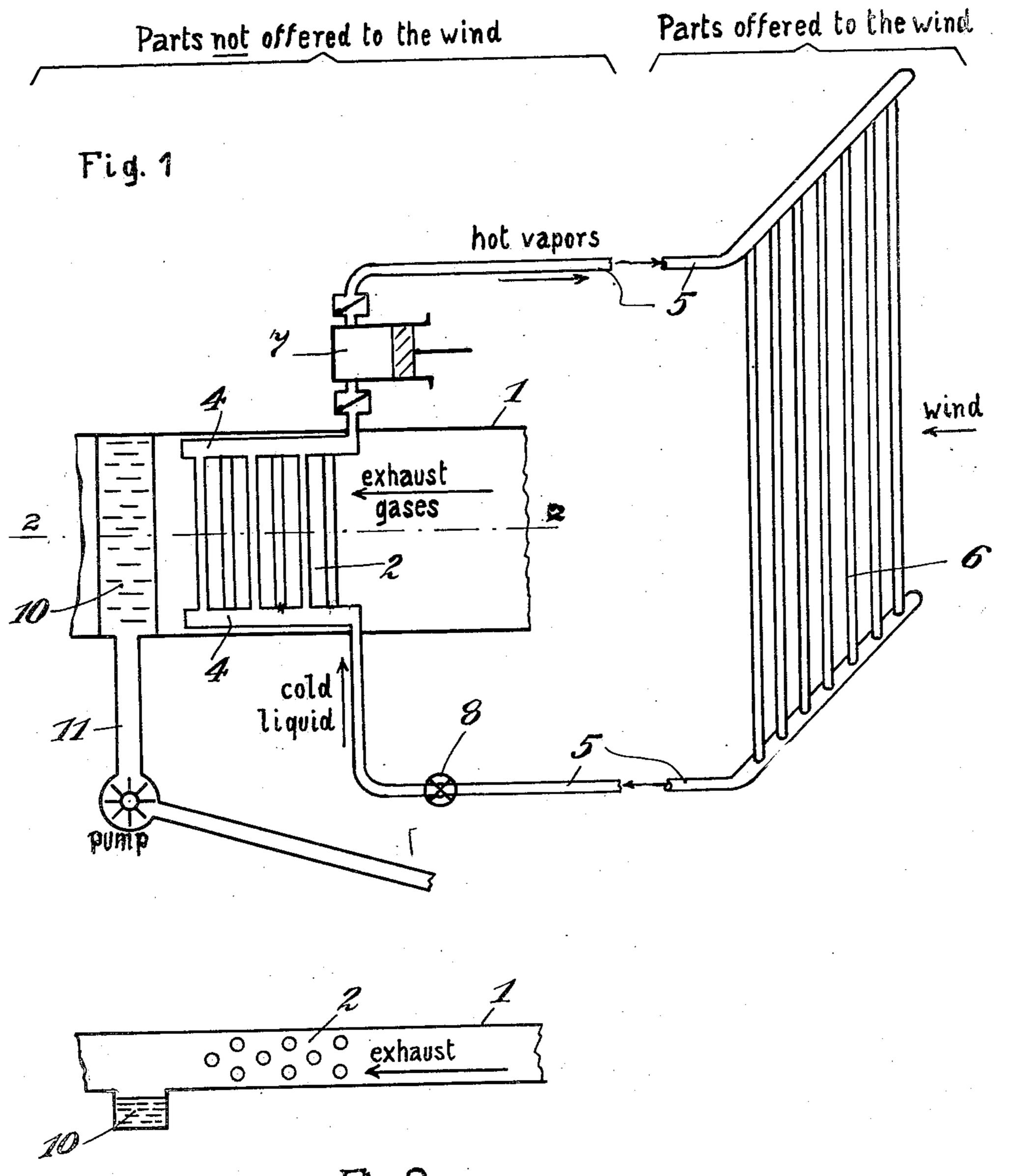


Fig.2

Paul Schroeder

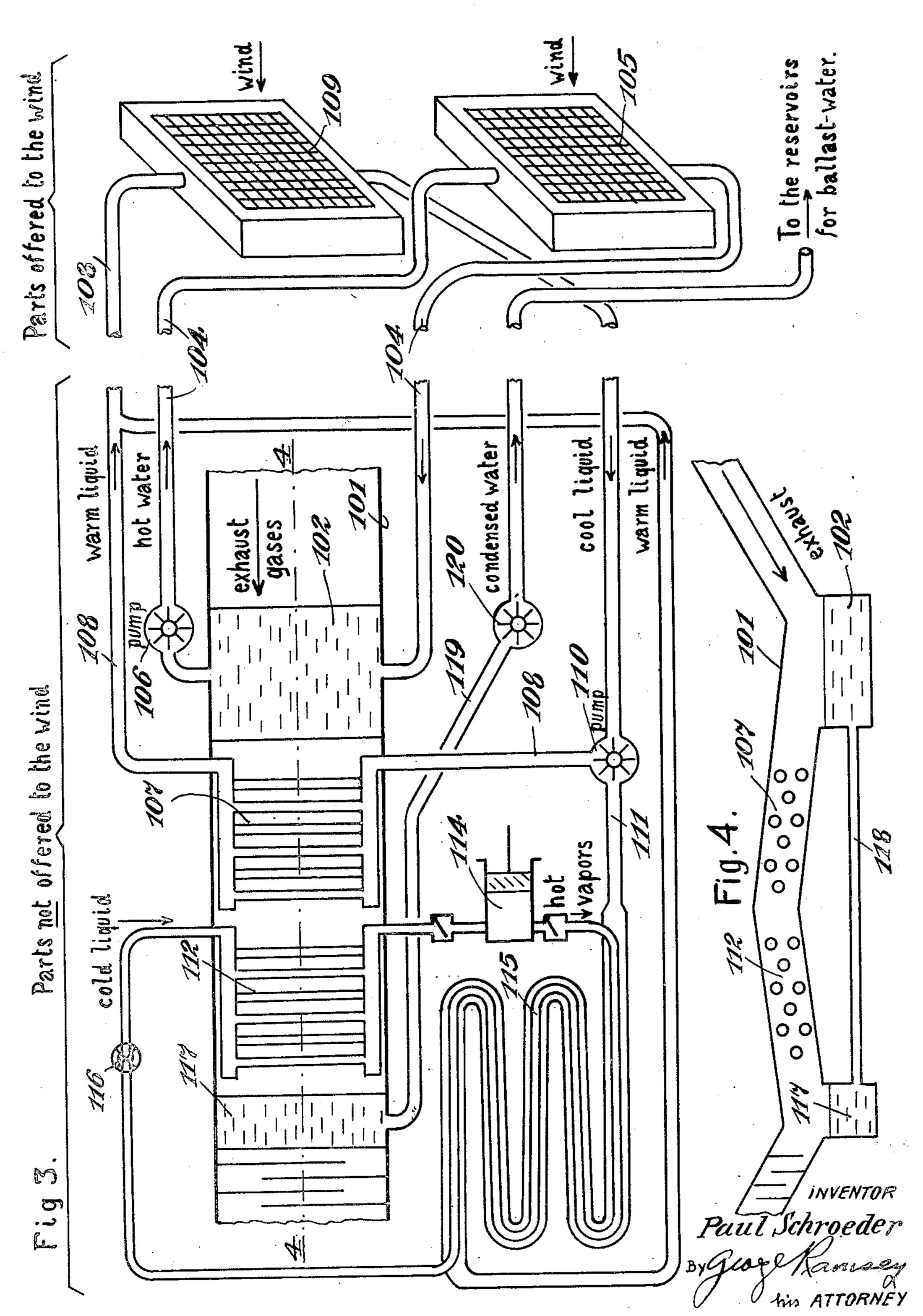
By George Rome,
his ATTORNEY

P. SCHRÖDER

CONDENSER FOR AIRSHIPS

Filed June 4, 1926

3 Sheets-Sheet 2



P. SCHRÖDER

CONDENSER FOR AIRSHIPS

3 Sheets-Sheet 3 Filed June 4, 1926 the wind 2 Parks offered 0 Ti qvid liquid 9 water ្រាធារា ીં લુળાં લે nem in condensed gases warm 1000 ダダア語 **50** 2000 /o o/ the wind Fig. 6. څ 0 0 offered Ιò 0 Parks not liguid cold 0 0 0 0 0 ਠੀ

UNITED STATES PATENT OFFICE.

PAUL SCHRÖDER, OF STUTTGART, GERMANY, ASSIGNOR TO EXCAELUM CORPORATION, A CORPORATION OF NEW YORK.

CONDENSER FOR AIRSHIPS.

Application filed June 4, 1926. Serial No. 113,616.

lighter-than-air airships, and more especially denser tubes comprising a grid set across the to method and apparatus for condensing exhaust conduit in such manner as to be in water vapor in the exhaust gases from in- the path of the exhaust gases. This con-

ship.

weight in flight due to fuel consumption. It sion valve, and a compressor. The refrigeris known that this loss of weight may be ant medium may pass through the expansion 10 compensated by condensing the moisture valve and expand in the condenser tubes in 65 present in the exhaust gases from the inter- such manner as to absorb heat from exhaust nal combustion engines used in propelling gases and then be drawn off through the the ship. In the methods heretofore pro- compressor and forced under pressure into posed for this purpose, the exhaust pipes the radiating system exposed to the air. 15 have been subjected to the direct cooling ef- whereby the heat taken up from the exhaust 70 fect of the air around the airship. This gases is dissipated to the atmosphere. By prior method has been impractical because means of this arrangement, it is possible to of the abnormal stresses set up in the exhaust cool the exhaust gases many degrees below pipes due to large changes in temperature. that of the surrounding atmosphere and 20 Furthermore, the direct cooling of exhaust thereby conserve the water vapor which is 75 upon the side walls of these pipes, thereby would be lost. tending to clog and choke the pipes and at It is recognized that the present invention 25 cooling operations, because the coating tions other than those specifically herein dis- 80 Another serious disadvantage of the prior be considered as illustrative and not in the systems is the fact that the air surrounding the ship may be of such a temperature that 30 the exhaust gases cannot be effectually air cooled, and therefore, the water vapor in these gases is lost. This is particularly liable to happen where the ship is flying over hot arid areas, such as deserts, where it is most 35 important and desirable to conserve all water

possible. The present invention overcomes the difficulties of the known prior art by utilizing an intermediate cooling fluid for purposes 40 of heat transfer, which fluid receives the heat from the exhaust gases and dissipates it through air cooled radiators. This fluid should be sufficiently volatile to permit the fluid to be liquefied by compression so that

The present invention relates broadly to tion, such gases may be passed through con-5 ternal combustion engines used to propel the denser may be connected in a closed circuit 60 comprising an air cooled radiator exposed Lighter-than-air airships gradually lose to the air outside of the airship, an expanpipes tends to cause soot to be deposited present in the exhaust gas which otherwise

the same time to greatly interfere with the may be embodied in apparatus and construcformed on the pipes acts as a heat insulator. closed so that the disclosure herewith is to

limiting sense.

Fig. 1 diagrammatically illustrates a sectional plan view of a simple form of appa- 85 ratus for carrying out the present invention. Fig. 2 is a diagrammatic sectional view on

line 2—2 of Fig. 1.

Fig. 3 is a sectional plan view which diagrammatically illustrates a multi-stage sys- 90 tem of cooling the exhaust gases.

Fig. 4 is a diagrammatic sectional view on

line 4—4 of Fig. 3.

Fig. 5 also diagrammatically illustrates a different multi-stage cooling arrangement of from that shown in Figs. 3 and 4.

Fig. 6 is a diagrammatic sectional view

taken on line 6—6 of Fig. 5.

Referring now more especially to Figs. 1 45 it may be again vaporized and the heat re- and 2, the exhaust conduit 1 is crossed by 100 quired for evaporation may be withdrawn condenser tubes 2 which are attached to from the hot exhaust gases. There are fluids suitable headers 4. These headers are conwhich in a liquid state evaporate readily at nected by suitable pipes 5 with an air ratemperatures considerably lower than that diator 6 exposed to the air surrounding the 50 required to properly cool the exhaust gases ship. The hot pipe line leading from the 105 to obtain condensation of the water therein, condenser 2 may be provided with a comand which may readily be reconverted to pressor 7, adapted to withdraw vaporized reliquids by pressure and the withdrawal of frigerant from the condenser and force the heat. Examples of such liquids are NH₃, same under pressure into a radiator 6. The SO₂, CO₂, CH₃CL. According to my inventool pipe line 5 leading to the condenser 110

from the radiator in which the refrigerant porize some of the water so that the gases expansion valve 8 adapted to control the expansion of the cooling medium as it enters 5 the condenser. Preferably, the medium used under pressure at a relatively high temperature and will assume a gaseous form when perature, where necessary, and the water in the exhaust gases, may be condensed and

for ballast, or as desired.

ated by the pump 120. 20 and 4, a more comprehensive cooling sys- a further modification of the present inven- 85 25 is connected by suitable pipe lines 104 to an The exhaust conduit 201 is crossed by a plu- 90 30 next encounter a condenser 107 which is like-ship, and the circulation of the cooling liq-95 35 cated outside of the airship. One of the diator 209 located outside of the ship. A 100 40 the pipe 108 carrying the warm fluid to the circulating pipe to the warm circulating 105 45 the exhaust gases. This third stage com- is connected in a closed system with a com- 110 116. The closed circuit for this final stage water troughs 216 and 217 are arranged in 115 liquefies at relatively high temperature un-55 able amount of heat when expanded in the condenser 112. A second trough 117 is also provided so that the water condensed from the exhaust gases will fall into either trough water may be equalized between the troughs. 102 or 117, and since these troughs are con- Preferably, baffle plates 221 are provided nected by compensating pipe 118, the excess beyond the refrigerating and cooling zones 125

is liquefied may be provided with a suitable tend to be very humid when they strike the cooling tubes further on in the exhaust conduit. This is desirable because it will cause a maximum amount of condensation on the 70 in this closed circuit is such as will liquefy cooling tubes with the result that the tubes will be drenched and deposited soot effectively washed off. Condensate may run dithe pressure is released, thereby taking up rectly back to reservoir 102, but in any event 10 the heat from its surroundings during the this reservoir will be kept properly filled 75 expansion cycle. By this means, the exhaust through equalizer pipe 118. A drain pipe gases may be cooled below atmospheric tem- 119 may be connected with one of the troughs, for example, the trough 117, to carry the condensed water to suitable reser-15 collected in the trough 10 from which it voirs (not shown) for storing the water as 80 may be removed by the drain pipe 11 and ballast. The flow of the water through the conveyed to suitable storage tanks to be used drain pipe 119 may be controlled or acceler-

Referring now more especially to Figs. 3 Figs. 5 and 6 diagrammatically illustrate tem for the exhaust gases is diagrammati- tion wherein closed systems of cooling mecally illustrated and comprises an exhaust diums provide multi-stages of cooling zones conduit 101 crossed by an open trough of for cooling the exhaust gases from internal preliminary cooling water 102, which trough combustion engines used to drive a ship. air cooled radiator 105. One of the pipe rality of condensers. A first condenser 202 lines may be provided with a suitable pump encountered by the exhaust gases is connect-106 to circulate the water through the trough ed by suitable circulating pipes 204 with an 102 and the radiator 105. The exhaust gases air radiator 205 located outside of the airwise connected by suitable pipes 108 with uid may be accelerated by a suitable pump an air cooled radiator 109 in such manner 206. The next stage of cooling comprises a as to take up the heat from the exhaust gases second condenser 207 likewise connected to and dissipate the same by the radiator lo-circulating pipes 208 with an air cooled rapipes 108 may likewise be provided with the pump 210 is adapted to circulate the cooling circulating pump 110, located preferably in liquid in this second stage cooling zone and the cool pipe, and a by-pass circuit 111 of may operate to drive the liquid through a cooled liquid may lead from the pump into by-pass pipe 211 which leads from the cool radiator. The by-pass circuit of cooled liq- pipe and the cool by-passed liquid may be uid may be utilized for cooling the liquid or used to cool a refrigerant used in the third medium in a refrigerating system compris- condenser 212, forming the final cooling zone ing the third cooling stage encountered by for the exhaust gases. This condenser 212 prises a condenser 112 connected in a closed pressor 214 for compressing the refrigerant circuit with a compressor 114, a cooling coil medium and an expansion valve 215 is pro-115, or the like, cooled by the by-pass circuit vided to control the expansion of the re-111 from the pump 110, and expansion valve frigerant into the condenser 212. Suitable of cooling preferably includes one of the vol- the exhaust conduit in such manner that the atile substances heretofore mentioned which condensed water will flow into the troughs and may be led to ballast reservoirs (not der pressure and which absorbs a consider- shown) by drain pipe 218 in which may be provided a pump 219. Preferably, the 120 troughs 216 and 217 are connected by compensating pipe 220 so that the condensed of water in one trough will flow through the in order that these plates may receive any pipe 118 into the other trough. Flow globules of water being carried along by the through this pipe may be needed to keep exhaust gases. The exhaust conduit may trough 102 filled. The hot gases, while be- be provided with inclined bottom portions ing cooled by the water in reservoir 102, va- constructed to drain the condensed water 130

in Fig. 6.

The present invention is designed more especially to produce an efficient cooling of 5 the exhaust gases and may be utilized where a ship is operating at a warm climate or heat from the cooling liquid to the outer air. warm atmosphere under which conditions air cooling alone may not sufficiently lower the exhaust gases of an internal combustion the temperature of the exhaust gases to engine propelling an airship, which com-10 properly condense the water content thereof. prises retaining the exhaust gases within The major portions of the system are lo- the airship, transferring heat from the excated within the airship to avoid added head haust gases to a vaporizing refrigerant, re- 75 resistance, and the only exposed parts are taining the refrigerant within the airship, the air cooled radiators which are exposed transferring heat from the refrigerant to a 15 to the atmosphere surrounding the ship.

1. In an airship driven by at least one 7. In an airship propelled by at least one 80 internal combustion engine, preliminary internal combustion engine, an exhaust concooling means comprising an air cooled radi-duit, a radiator outside of the airship adapt-20 ator adapted to contain a circulating liquid ed to contain a constantly liquid cooling adapted to dissipate a portion of the heat in fluid, means to conduct said fluid to and the engine exhaust gases to the surrounding from the radiator from a point within the 85 atmosphere, final cooling means adapted to airship, means to transfer heat from excontain an expansible substance adapted to haust gases in the conduit to said cooling 25 absorb heat while expanding, conduit means fluid to initially cool the gases, and means to direct said gases to both of said cooling to thereafter transfer heat from the initialmeans, and storage means for the condensate. ly cooled gases to a vaporizing refrigerant. 90

2. The method of condensing water vapors present in the exhaust gases from internal internal combustion engine; at least two 30 combustion engines driving an airship, com- radiators located outside of the airship, one prising preliminarily cooling said gases by for a preliminary cooling circuit and ore liquid cooling means, finally cooling said for a further cooling circuit; cooling cir gases by gasifying a volatile substance, and cuits comprising means to conduct constant dissipating the heat received by said sub- ly liquid cooling fluids to and from the radi-35 stance to the atmosphere surrounding said ators from points within the airship; ar

ship.

3. The method of condensing water vapors present in the exhaust gases of internal combustion engines driving an airship, compris-40 ing retaining the gases in conduits protected from the outside atmosphere, preliminarily cooling said gases by liquid cooling means, finally cooling said gases by gasifying a volatile substance, dissipating the heat re-45 ceived by said substance to the outside atmosphere surrounding said ship, and collecting and storing the condensate.

4. In an airship propelled by at least one internal combustion engine, an exhaust con-50 duit within the airship, a heat transfer surface positioned to receive heat from gases in the conduit, means to conduct a vaporizing refrigerant to and from said surface, means to compress the vaporized refrigerant, 55 means within the airship to transfer heat from the compressed refrigerant to a cooling liquid, and means to conduct the cooling liquid to a point outside of the airship and dissipate heat therefrom to the surrounding air.

5. The method of recovering water from the exhaust gases of an internal combustion engine propelling an airship, which comprises

into the troughs 216 and 217 as is illustrated retaining the exhaust gases in conduits located wholly within the airship, trans- 65 ferring heat from the gases to a vaporizing refrigerant, transferring heat from the refrigerant to a cooling liqud, and dissipating

6. The method of recovering water from 70 cooling liquid, and dissipating heat from Having described my invention, I claim:— the cooling liquid to the surrounding air.

8. In an airship propelled by at least one exhaust conduit within the airship, means in one zone along the exhaust conduit to 100 transfer heat from the exhaust gases to the cooling fluid of the preliminary cooling circuit; means in another zone along the exhaust circuit to transfer additional heat from the exhaust gases to a vaporizing refrigerant; and means within the airship to transfer heat from the refrigerant to the cooling liquid of the second mentioned cooling circuit.

9. In an airship propelled by at least one internal combustion engine, an exhaust conduit, means to condense moisture from gases flowing through said conduit by cooling the gases with a vaporizing refrigerant, and 115 means to humidify the gases before the moisture is condensed therefrom.

10. In an airship propelled by at least one internal combustion engine, an exhaust conduit located within the airship, a heat absorbing surface within the conduit, means to conduct a vaporizing refrigerant to and from said surface, and a water pan opening into said conduit on the upstream side of said heat absorbing surface.

PAUL SCHRÖDER.