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Fieber et al.

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(54) **FLUID DELIVERY SYSTEM FOR A FIRE APPARATUS**

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A62C 27/00 (2006.01)

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CPC **A62C 37/00** (2013.01); **A62C 5/002** (2013.01); **A62C 27/00** (2013.01)

(58) **Field of Classification Search**

CPC A62C 37/00; A62C 37/04; A62C 37/08; A62C 5/002; A62C 27/00

See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

4,324,294 A	4/1982	McLoughlin et al.
4,673,335 A *	6/1987	Nicodemus F04F 5/24 417/174
5,074,522 A	12/1991	Reynolds et al.
5,788,158 A	8/1998	Relyea
5,839,664 A	11/1998	Relyea
6,138,767 A	10/2000	La Vergne
6,659,187 B1	12/2003	Crabtree et al.

(Continued)

FOREIGN PATENT DOCUMENTS

DE	10 2013 201 299 A1	9/2016
EP	0 230 860 A1	8/1987

OTHER PUBLICATIONS

Notification of Transmittal of the International Search Report and the Written Opinion of the International Searching Authority, or the Declaration, Oshkosh Corporation, PCT/US2018/042124, Nov. 14, 2018.

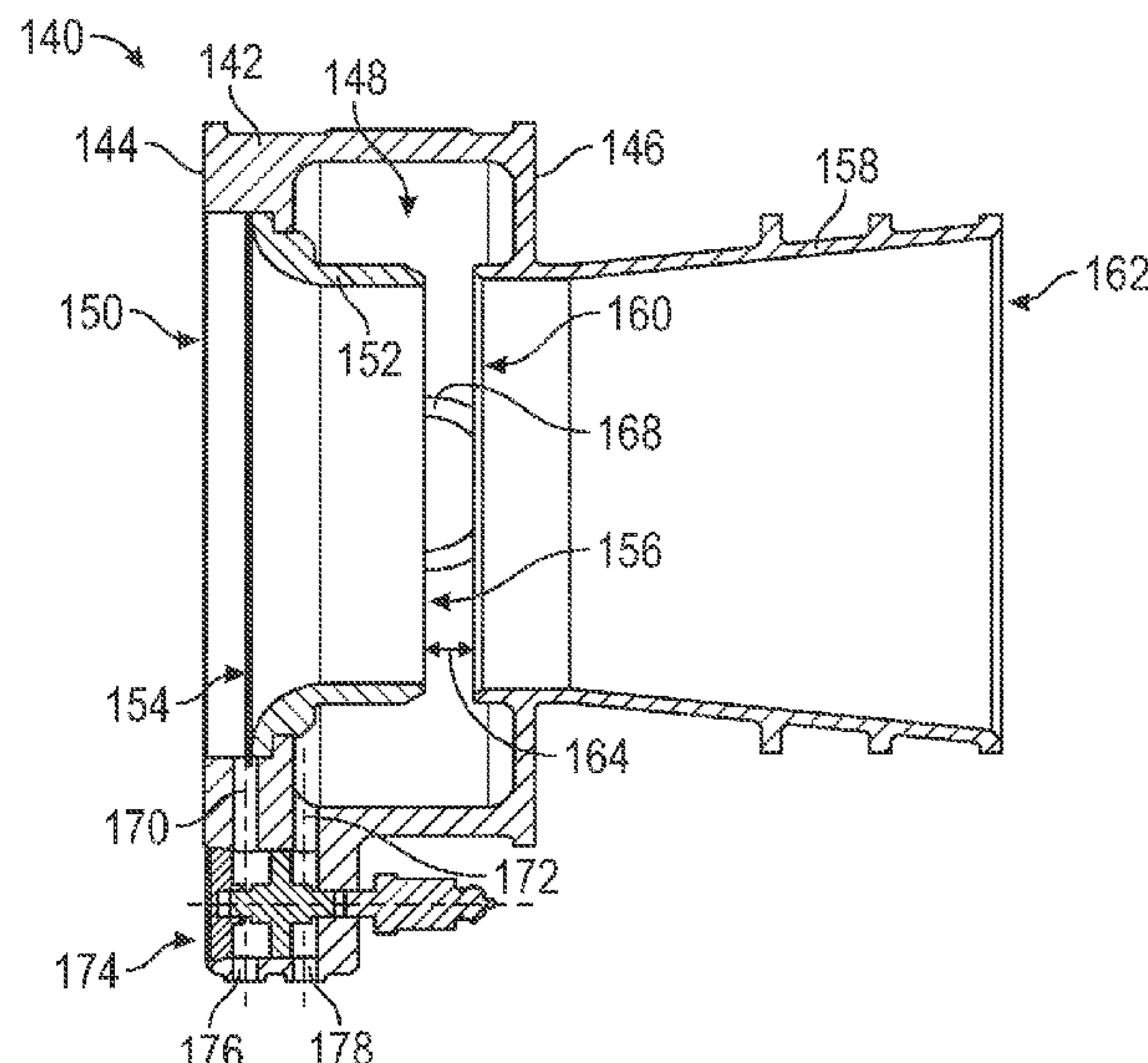
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(57) **ABSTRACT**

A ratio controller includes a housing defining a mixing chamber. The housing has a water inlet configured to receive water from a water supply, an agent inlet configured to receive agent from an agent supply, an outlet configured to output an agent-water solution, a first pressure port extending through a sidewall of the housing at a first position proximate the water inlet, and a second pressure port extending through the sidewall of the housing at a second position proximate the mixing chamber.

10 Claims, 17 Drawing Sheets

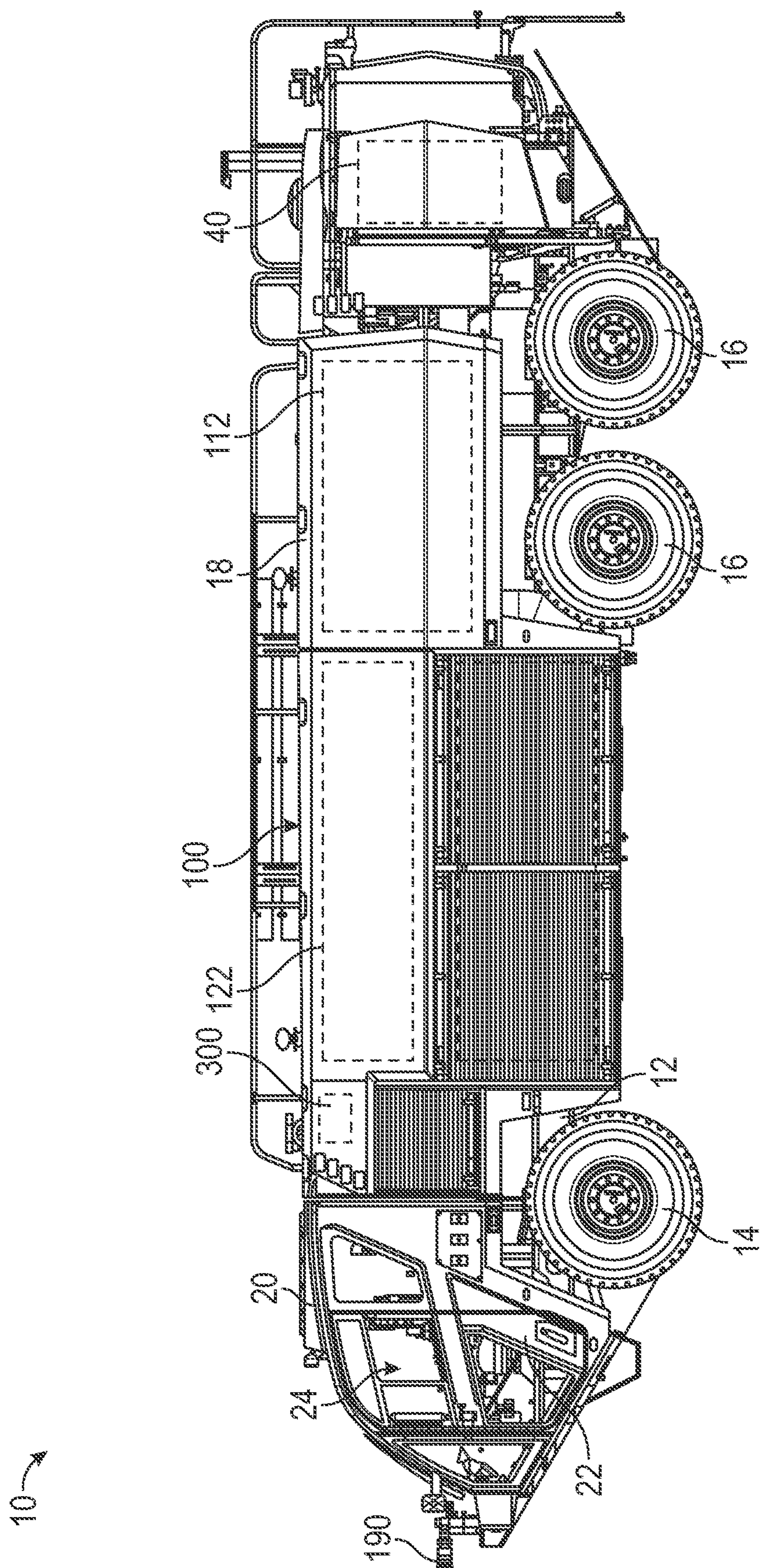


(56) **References Cited**

U.S. PATENT DOCUMENTS

6,860,332	B1	3/2005	Archer et al.	
7,140,552	B1 *	11/2006	Crabtree	A62C 31/02 169/44
7,234,534	B2	6/2007	Froland et al.	
7,331,586	B2	2/2008	Trinkner et al.	
7,367,361	B2 *	5/2008	Steingass	B01F 25/312 137/895
7,389,826	B2	6/2008	Linsmeier et al.	
7,611,075	B2	11/2009	Relyea et al.	
7,784,554	B2	8/2010	Grady et al.	
7,874,373	B2	1/2011	Morrow et al.	
7,987,916	B2	8/2011	Laskaris et al.	
8,307,907	B2	11/2012	Laskaris et al.	
8,376,719	B2	2/2013	Grady et al.	
8,555,625	B2	10/2013	Trinkner et al.	
8,739,892	B2	6/2014	Moore et al.	
8,801,393	B2	8/2014	Crabtree et al.	
9,061,169	B2	6/2015	Linsmeier	
9,149,671	B2	10/2015	Hosfield et al.	
9,327,150	B2	5/2016	Moore et al.	
9,504,863	B2	11/2016	Moore	
9,555,273	B2	1/2017	Stephens et al.	
9,579,530	B2	2/2017	Betz et al.	
9,597,536	B1	3/2017	Moore	
9,814,915	B2	11/2017	Moore	
11,185,728	B2	11/2021	Fieber et al.	
2016/0346576	A1	12/2016	Hammer	
2017/0009904	A1	1/2017	Mcloughlin	

* cited by examiner







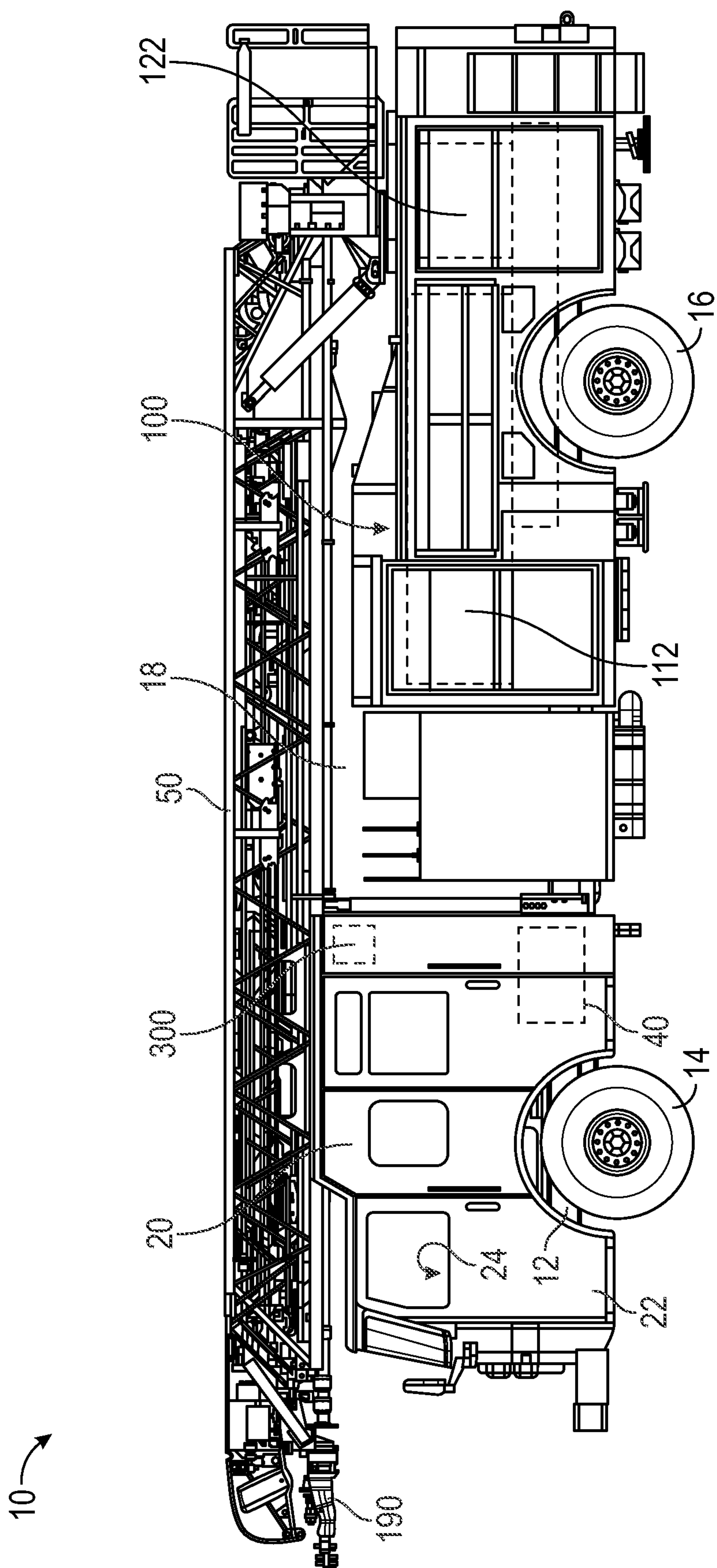



FIG. 2

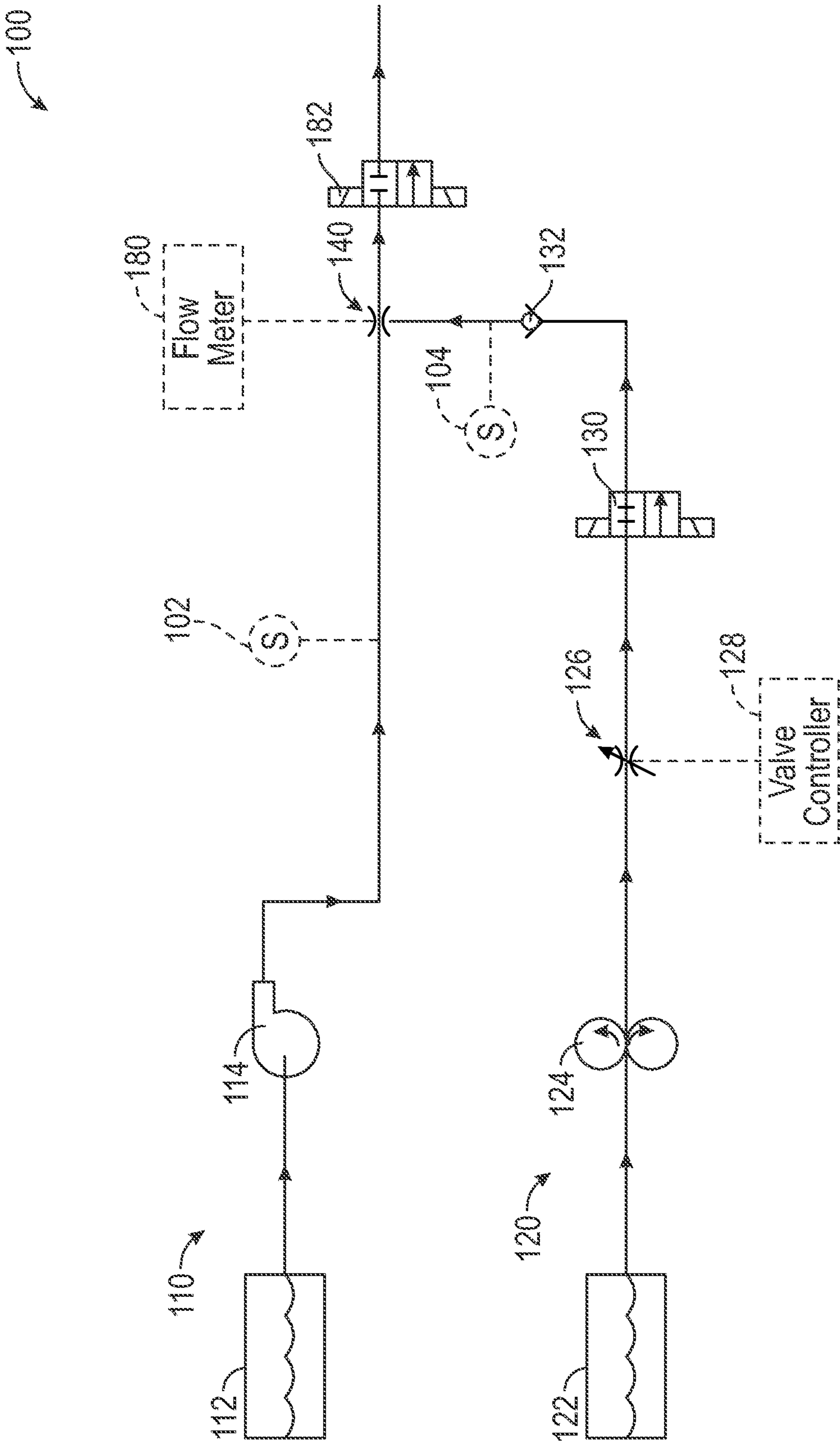
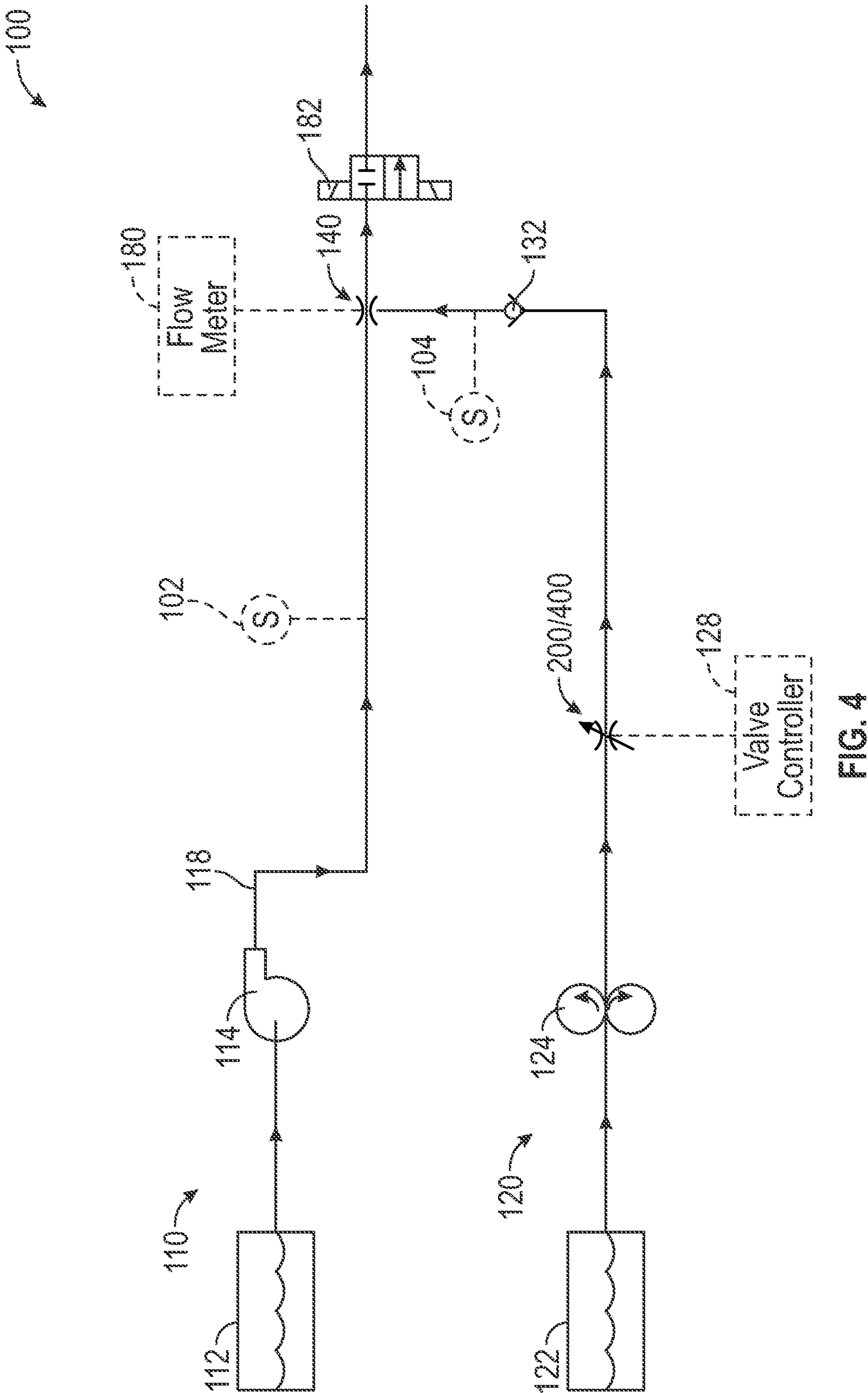


FIG. 3



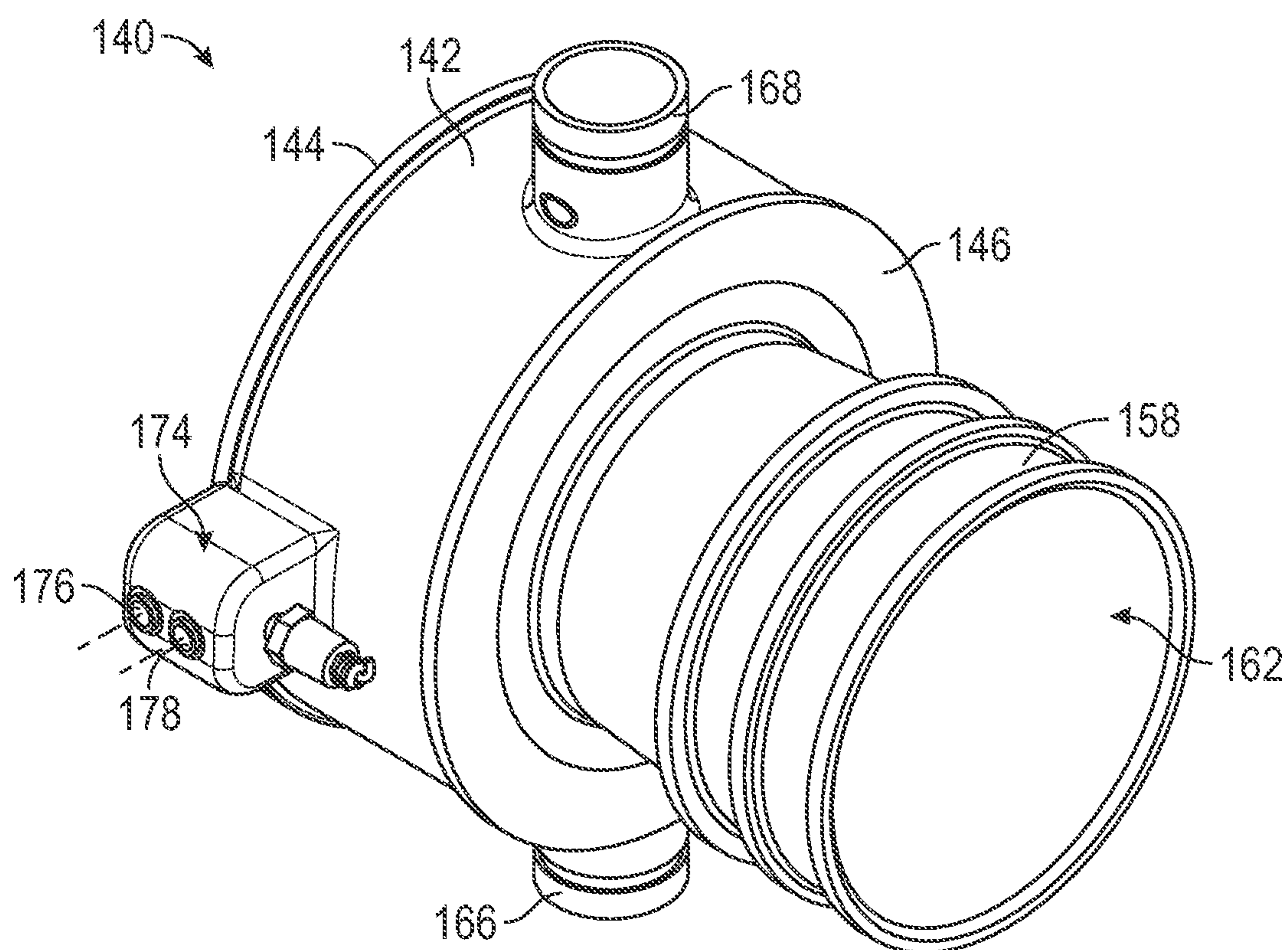


FIG. 5A

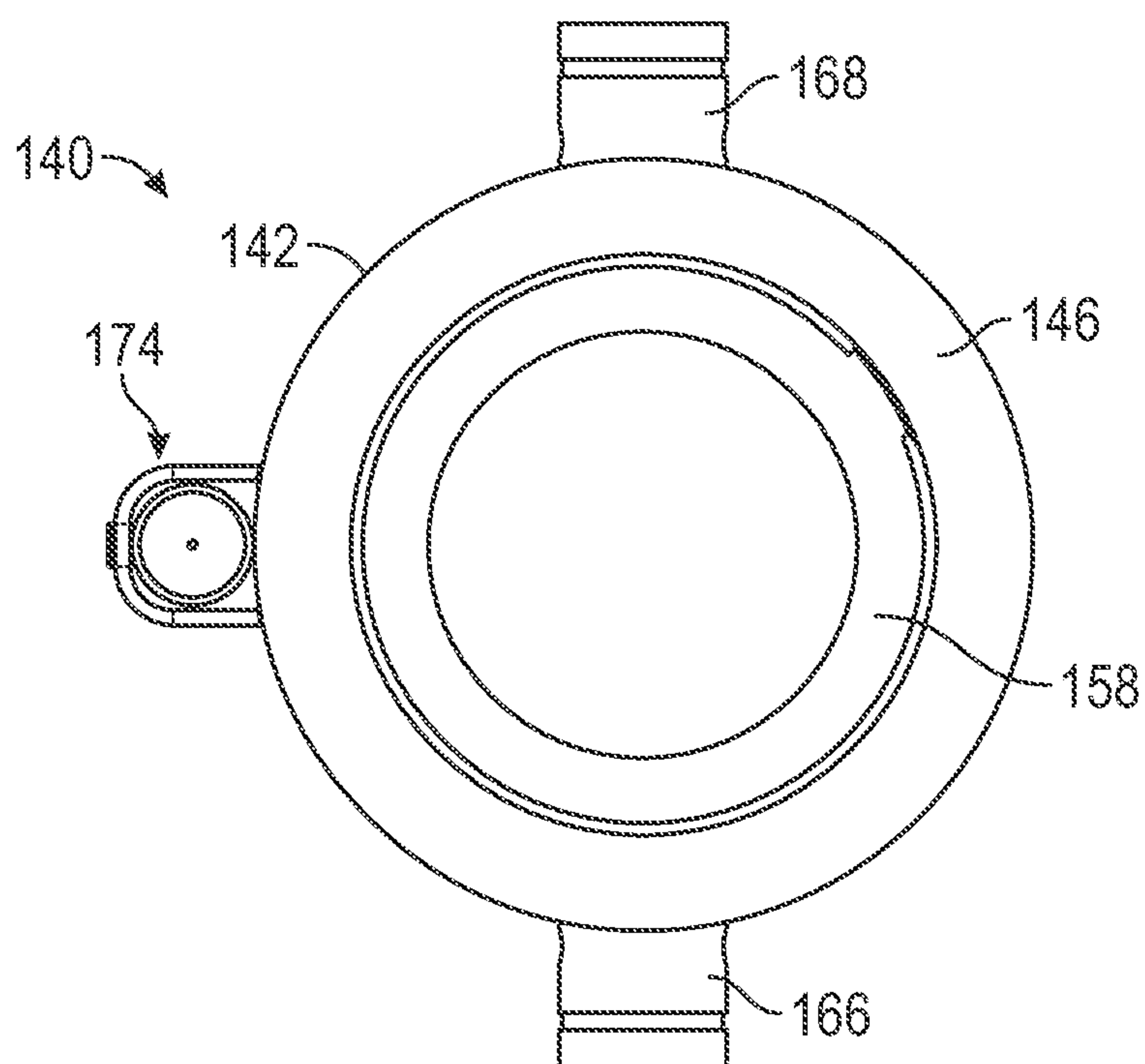


FIG. 5B

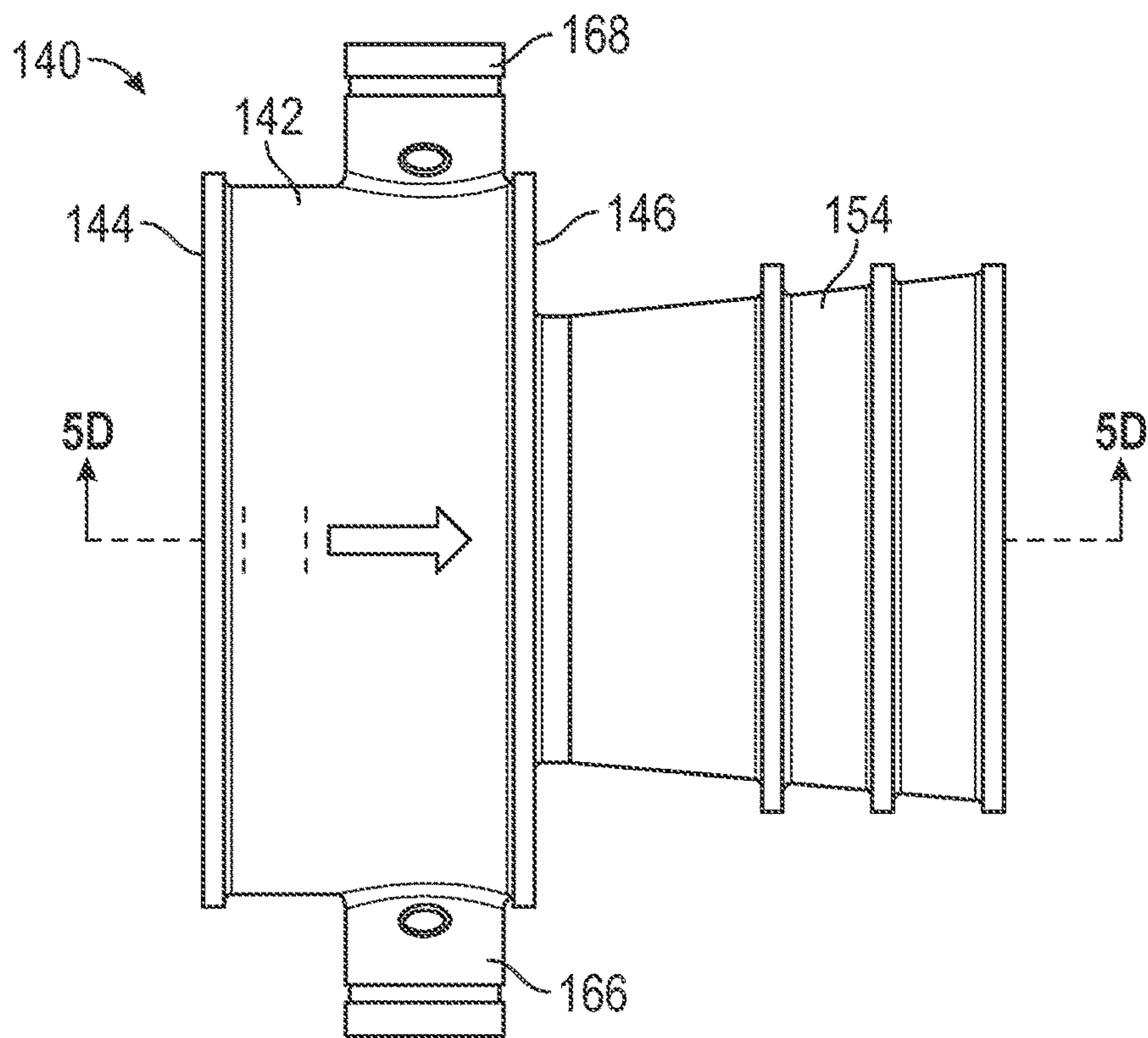


FIG. 5C

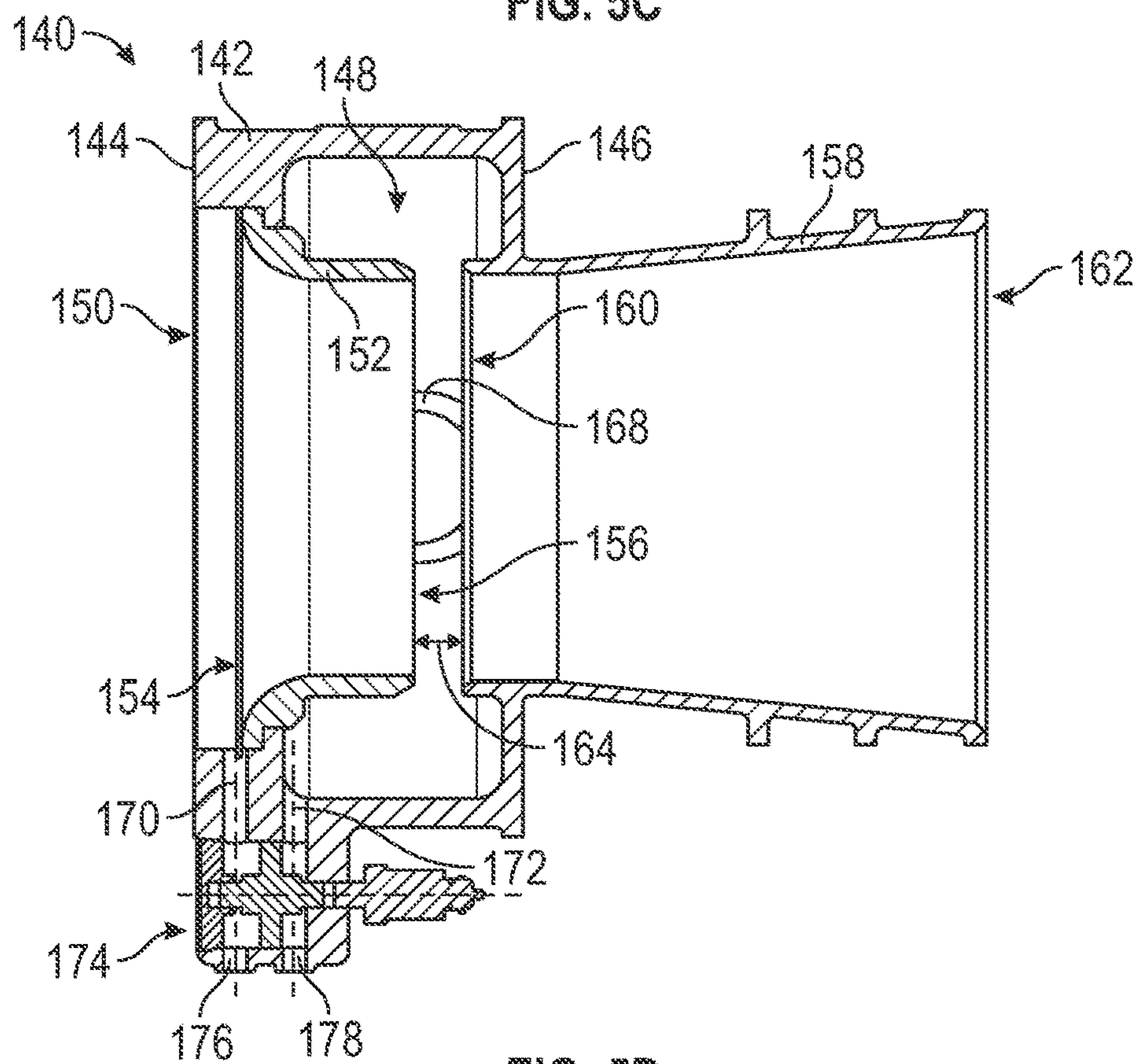
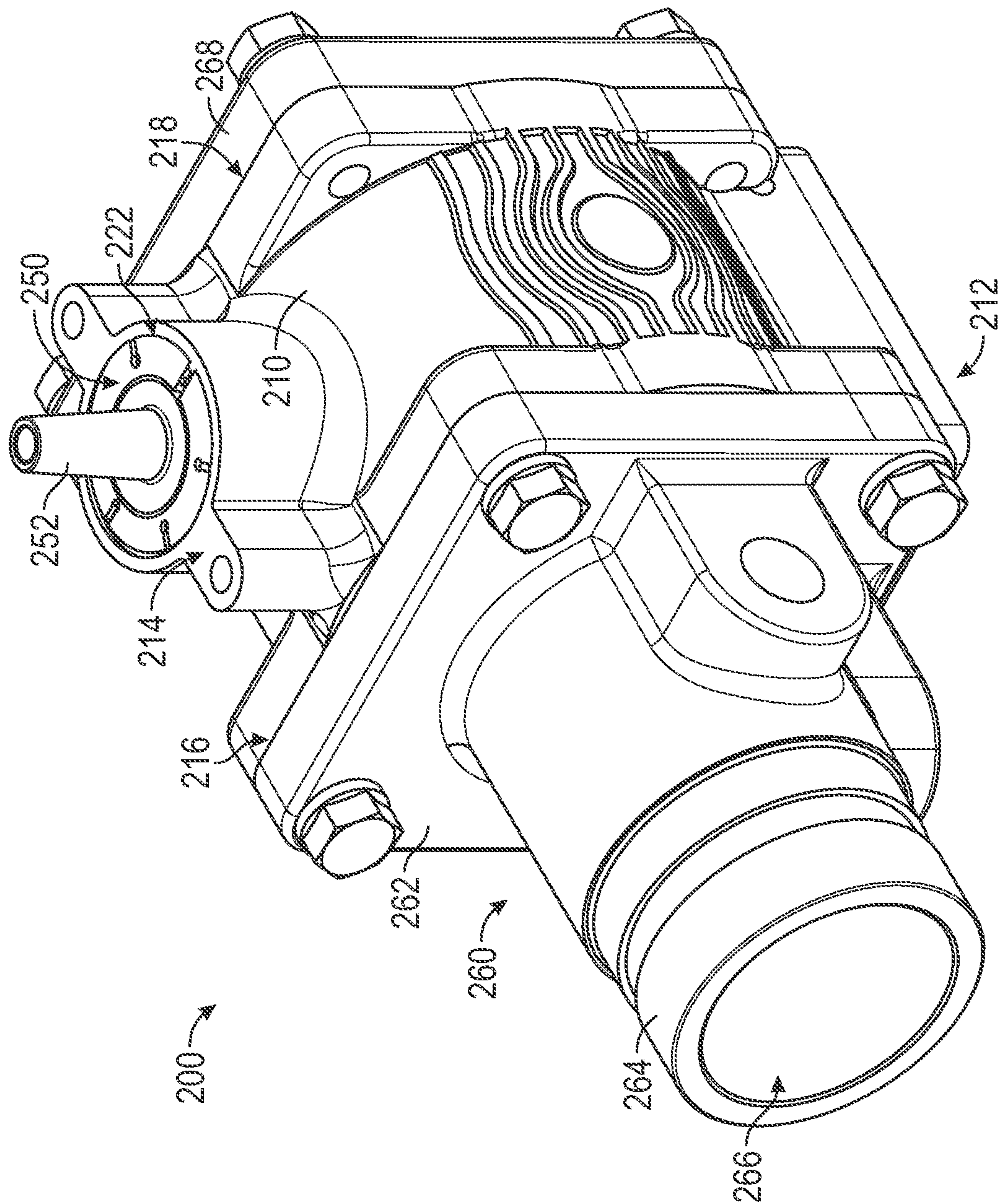


FIG. 5D



WELL

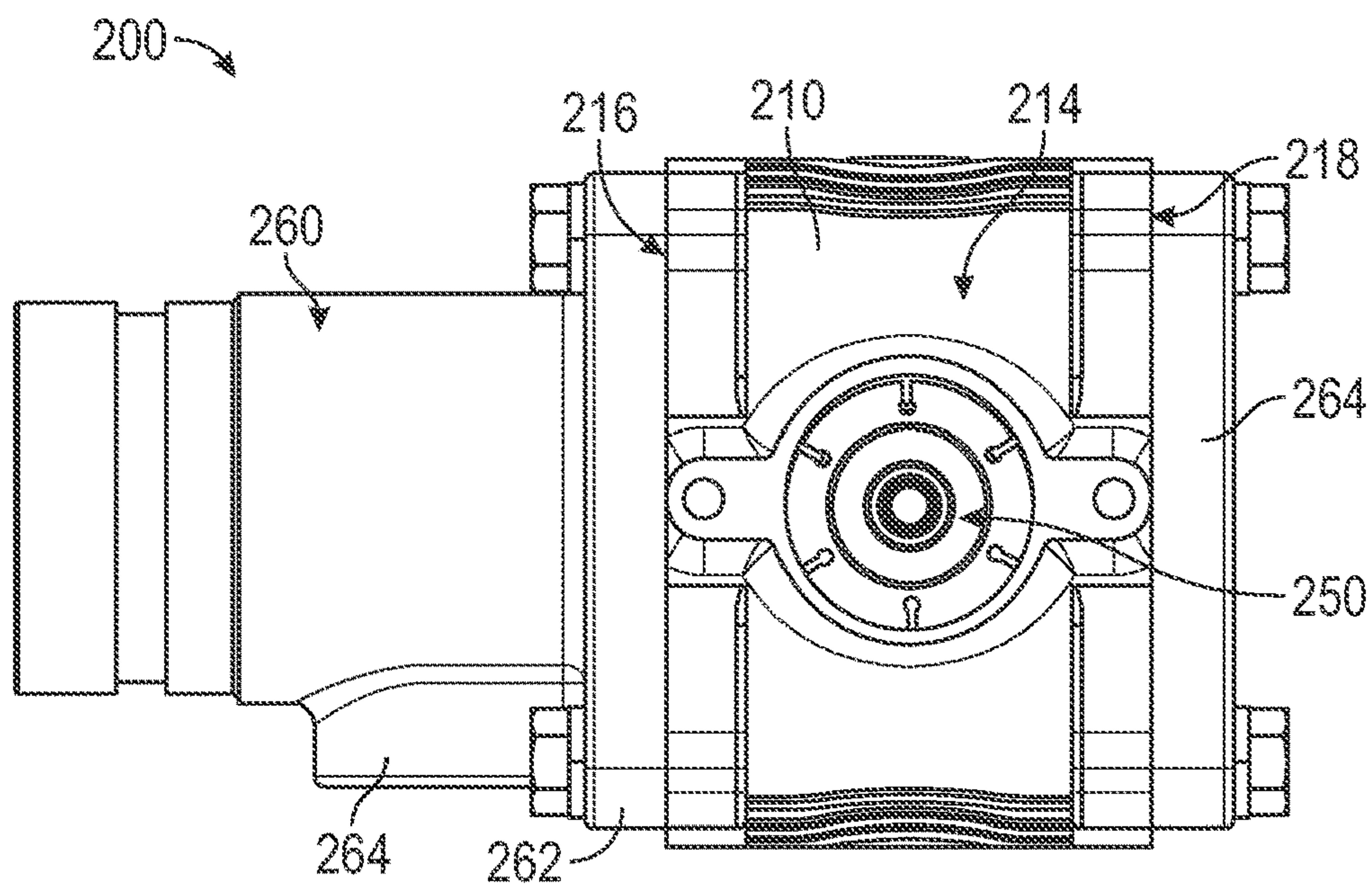


FIG. 6B

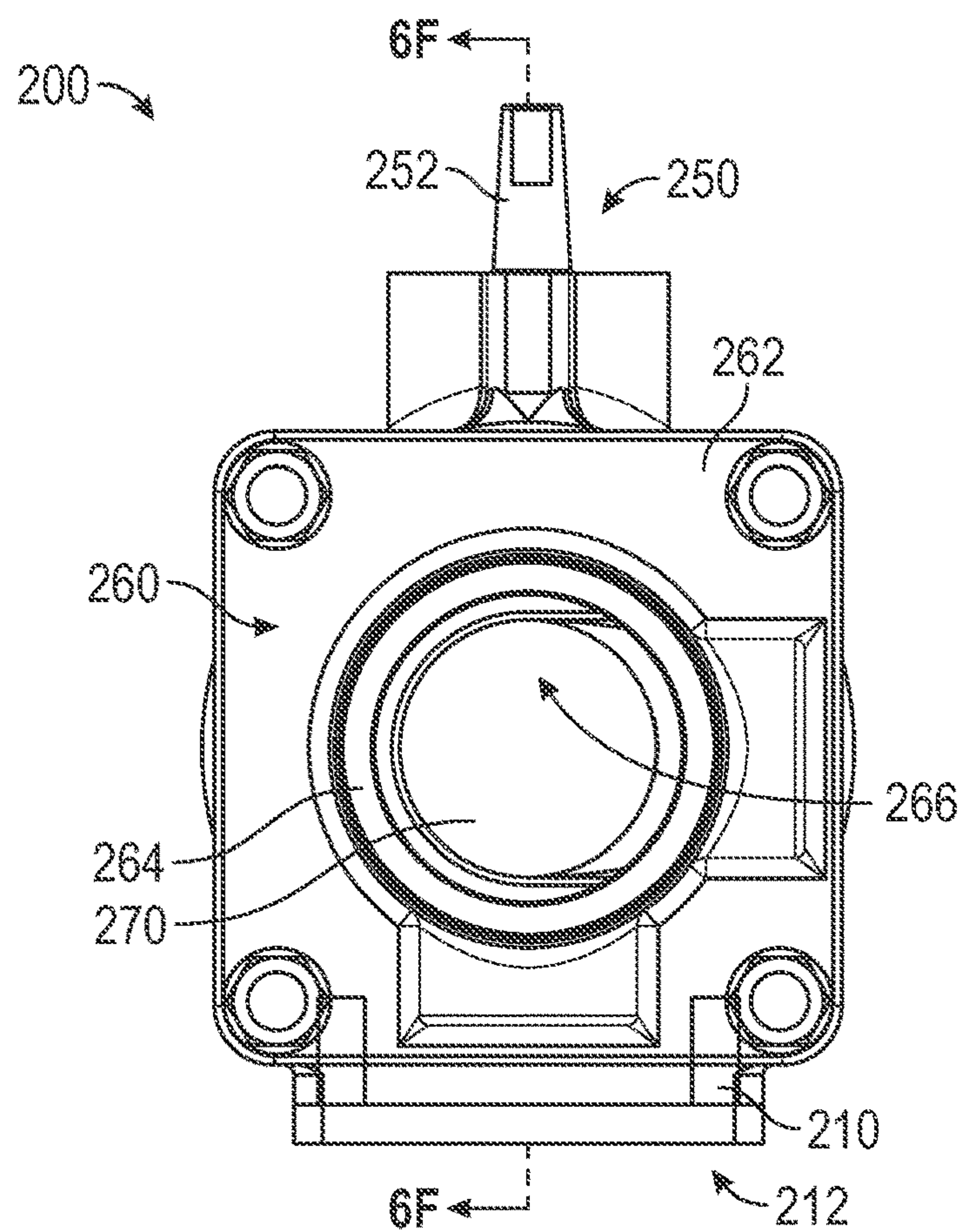


FIG. 6C

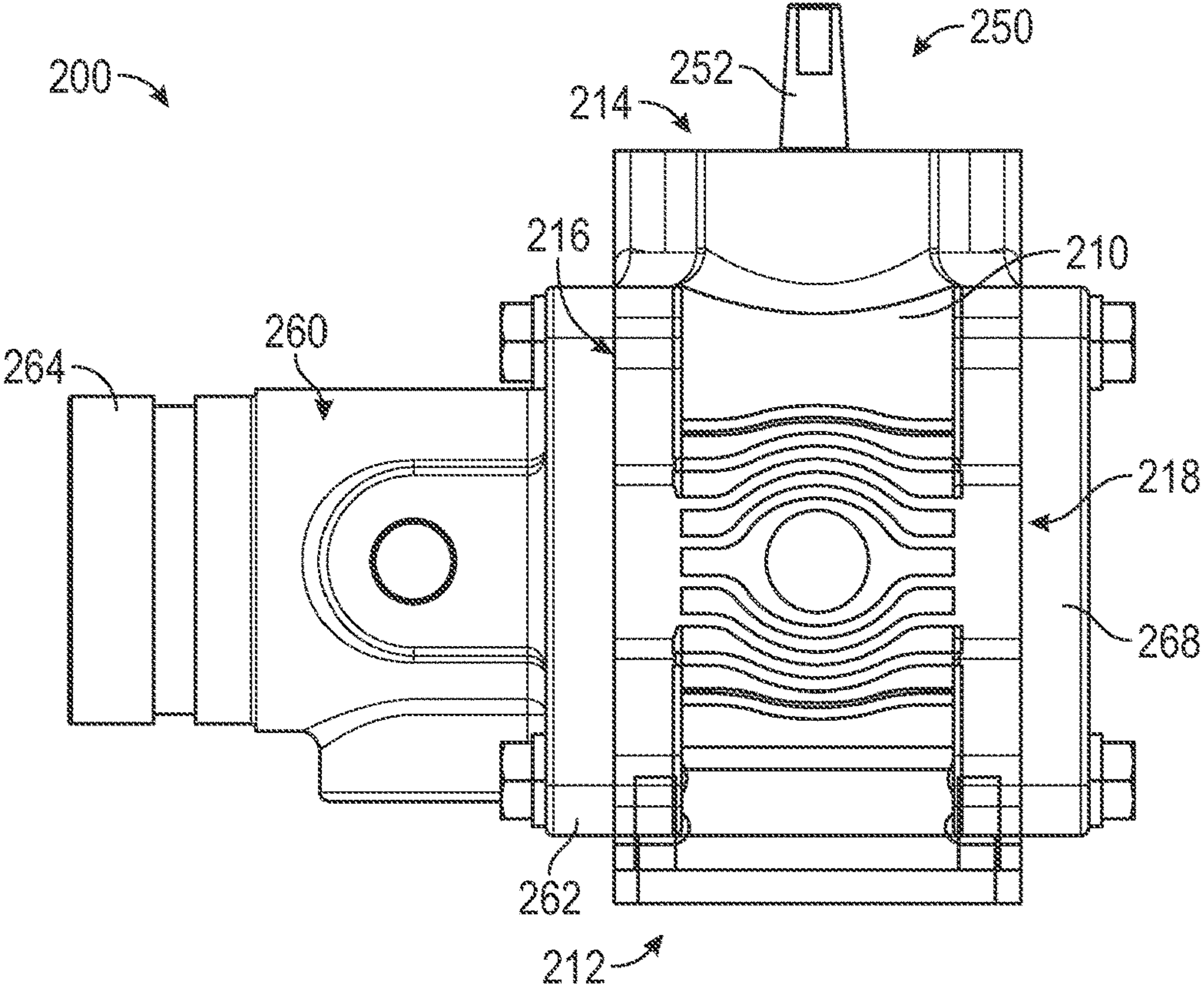


FIG. 6D

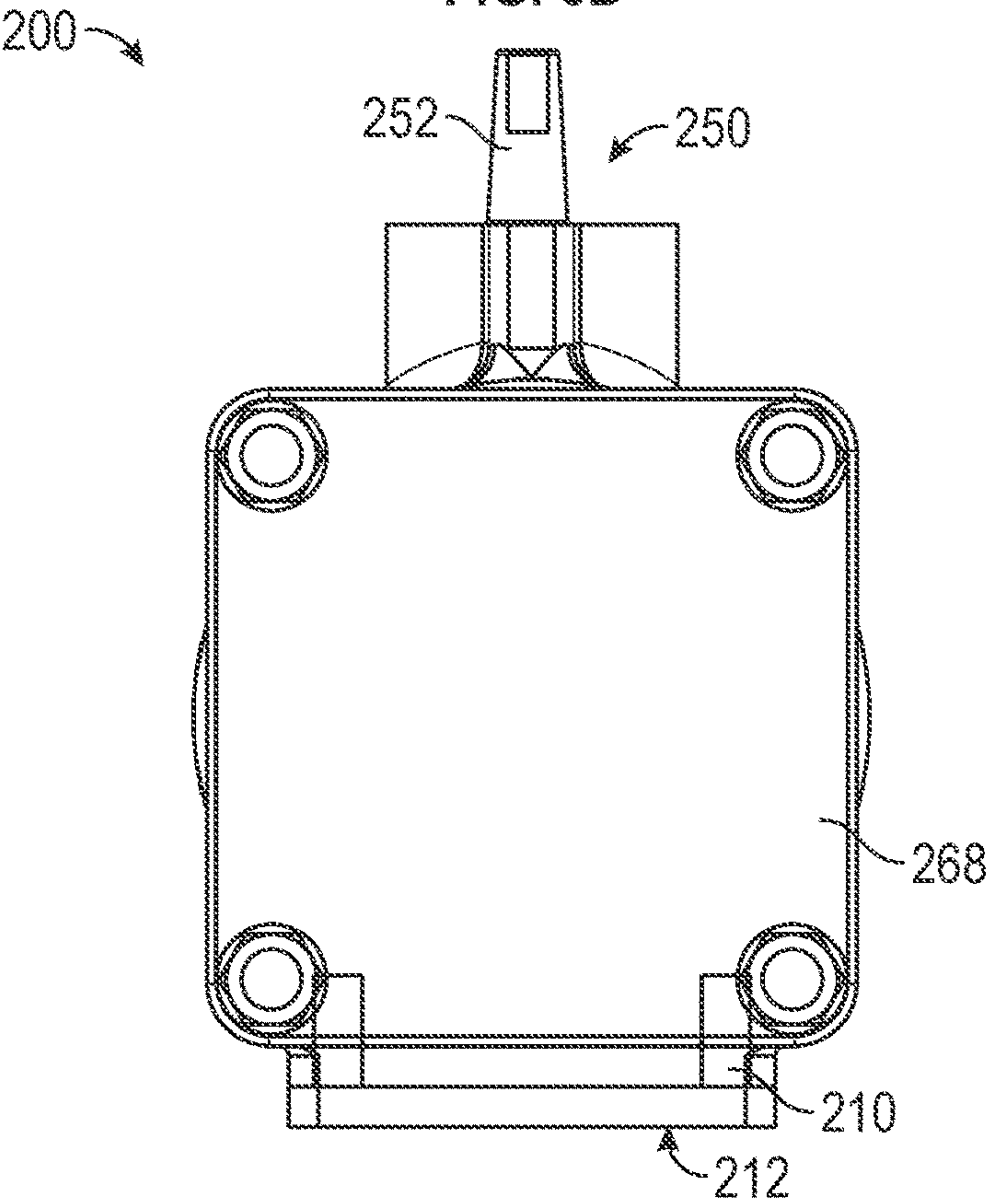


FIG. 6E

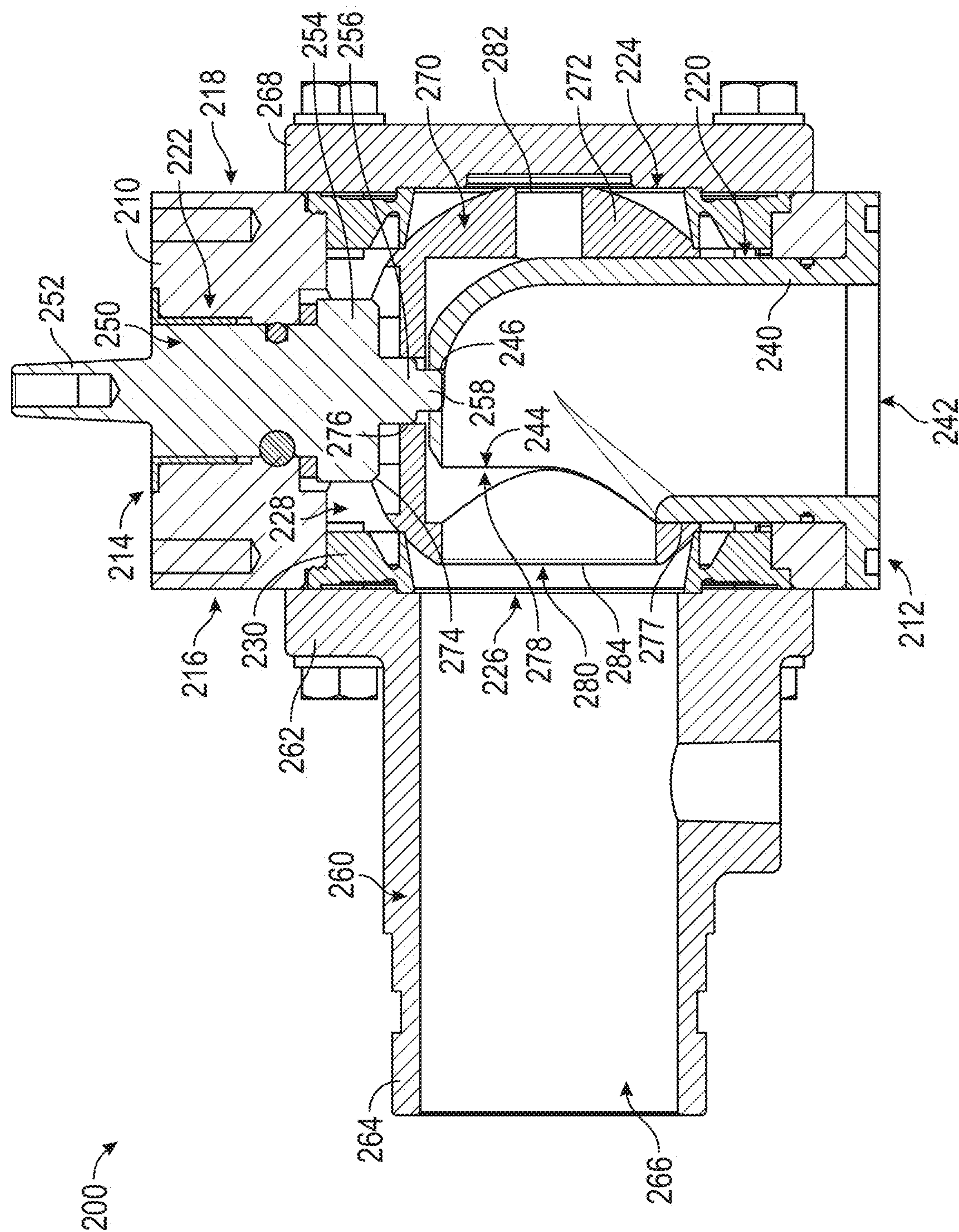


FIG. 6F

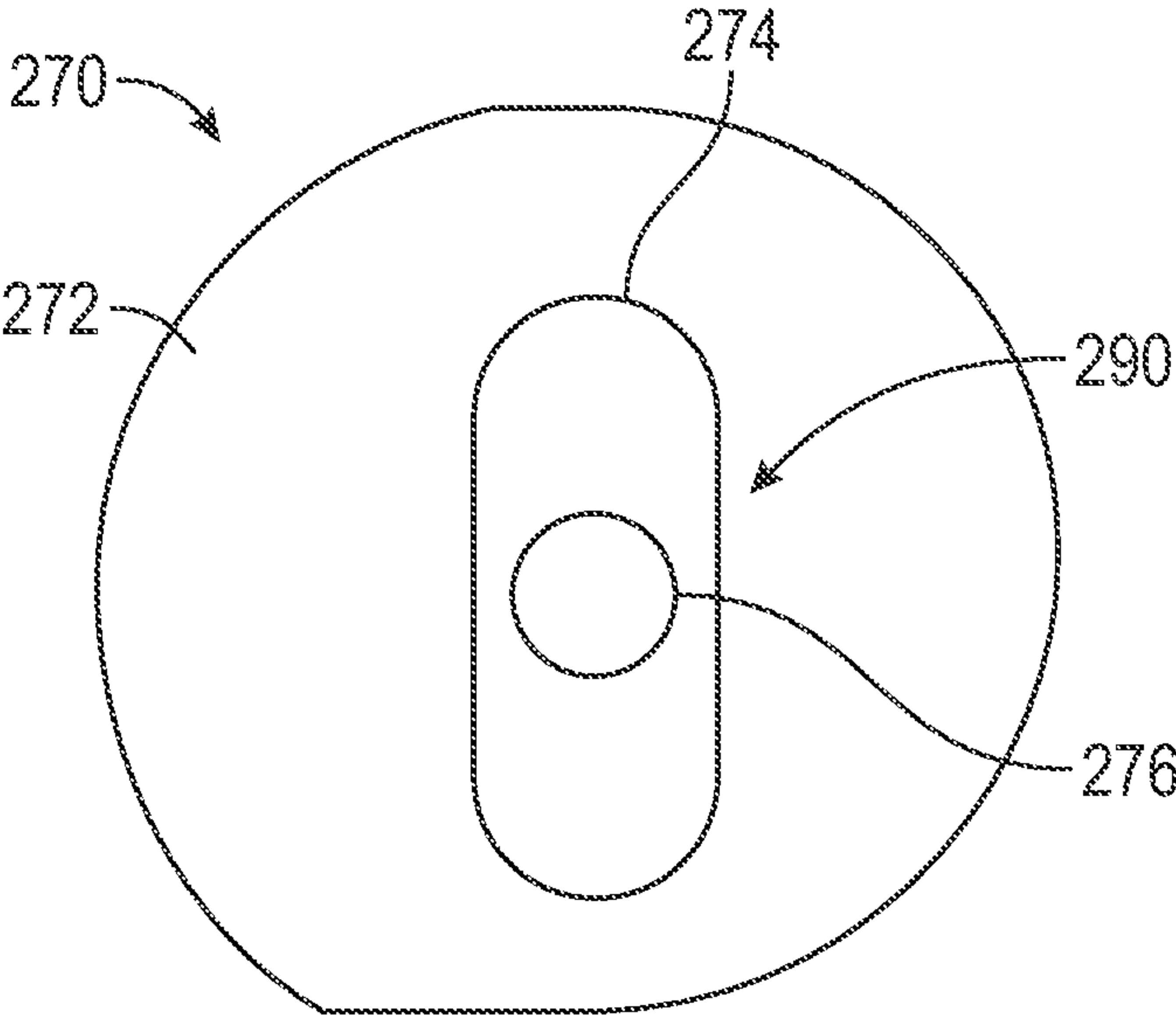


FIG. 7A

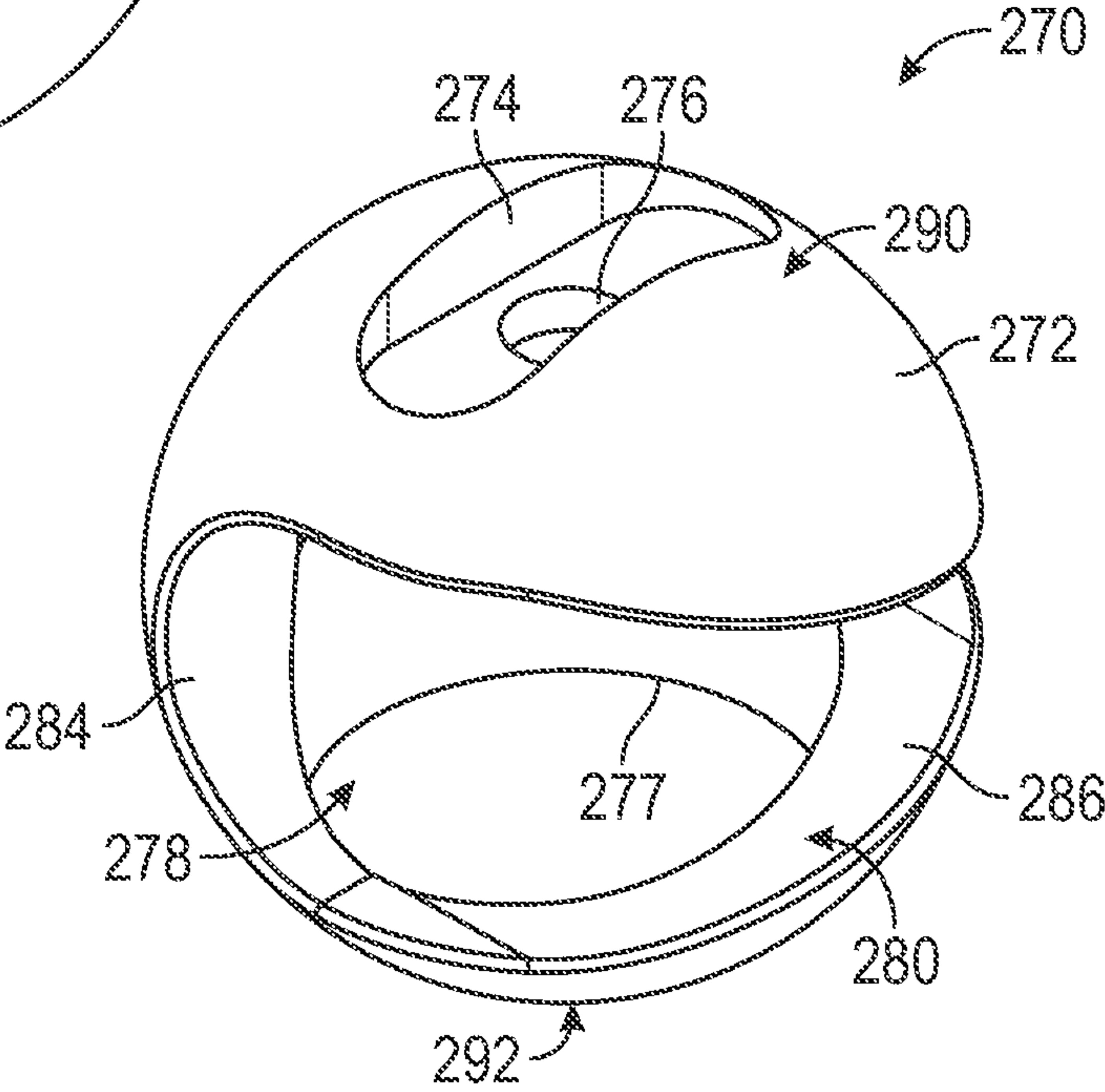


FIG. 7B

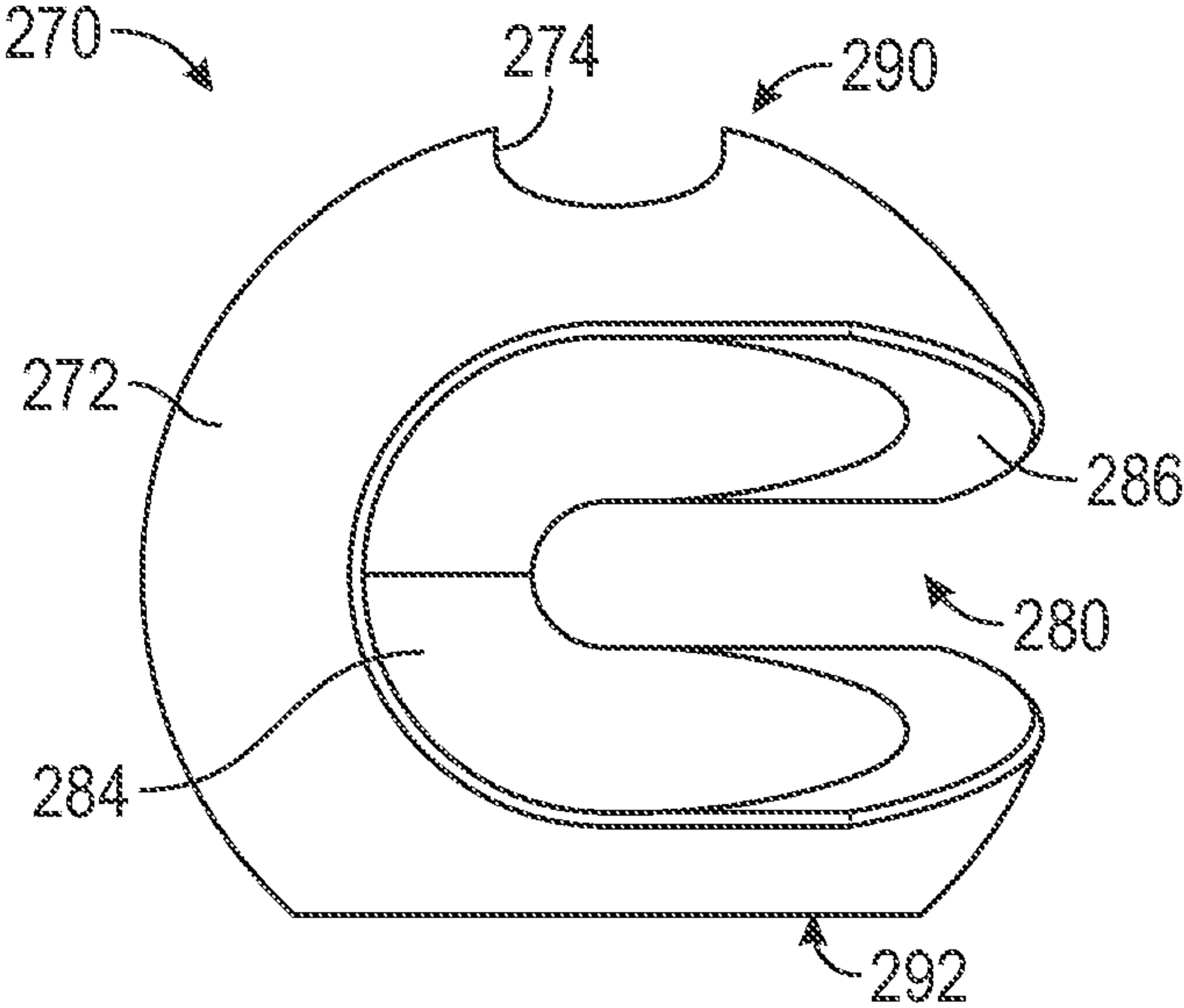


FIG. 7C

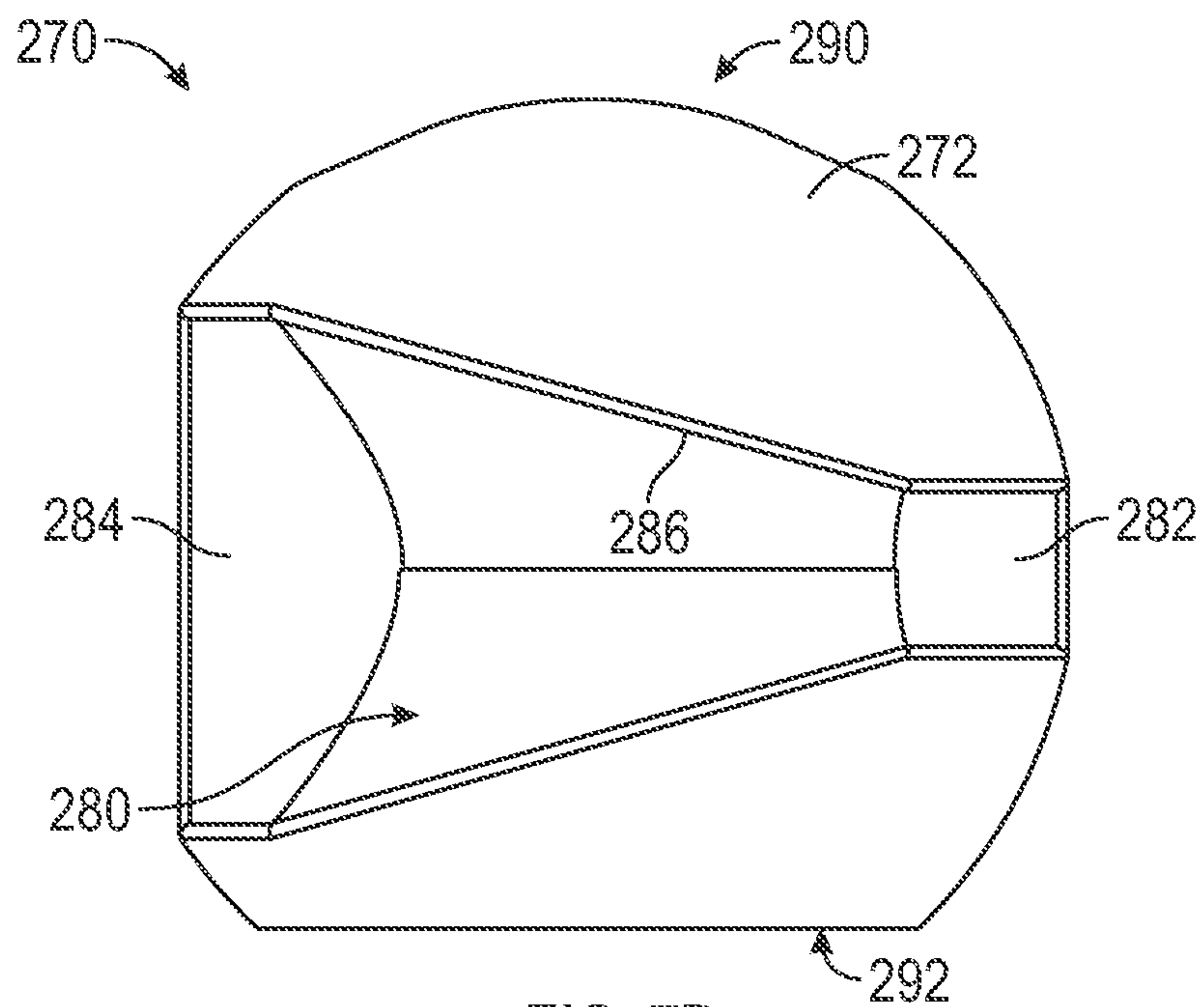


FIG. 7D

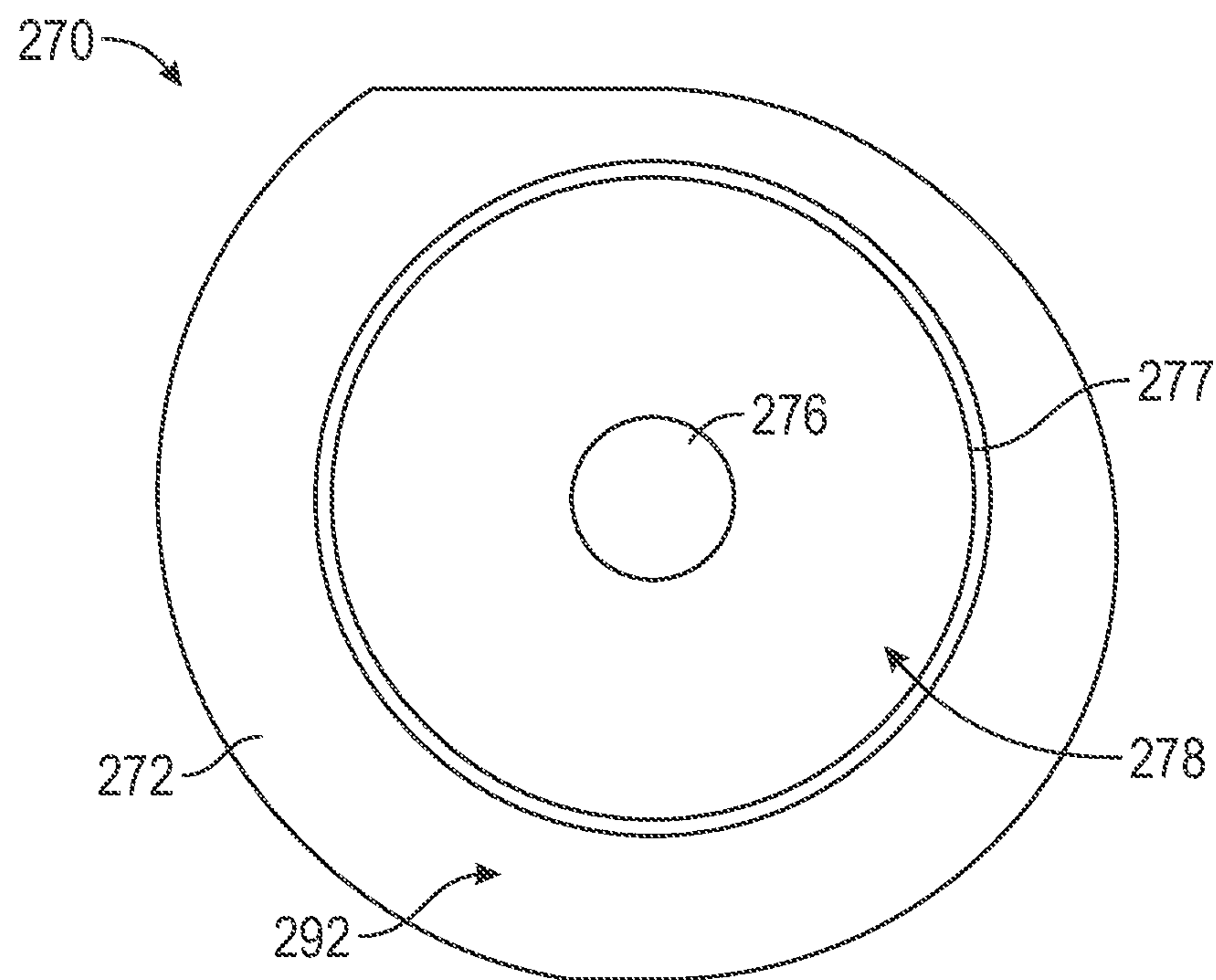


FIG. 7E

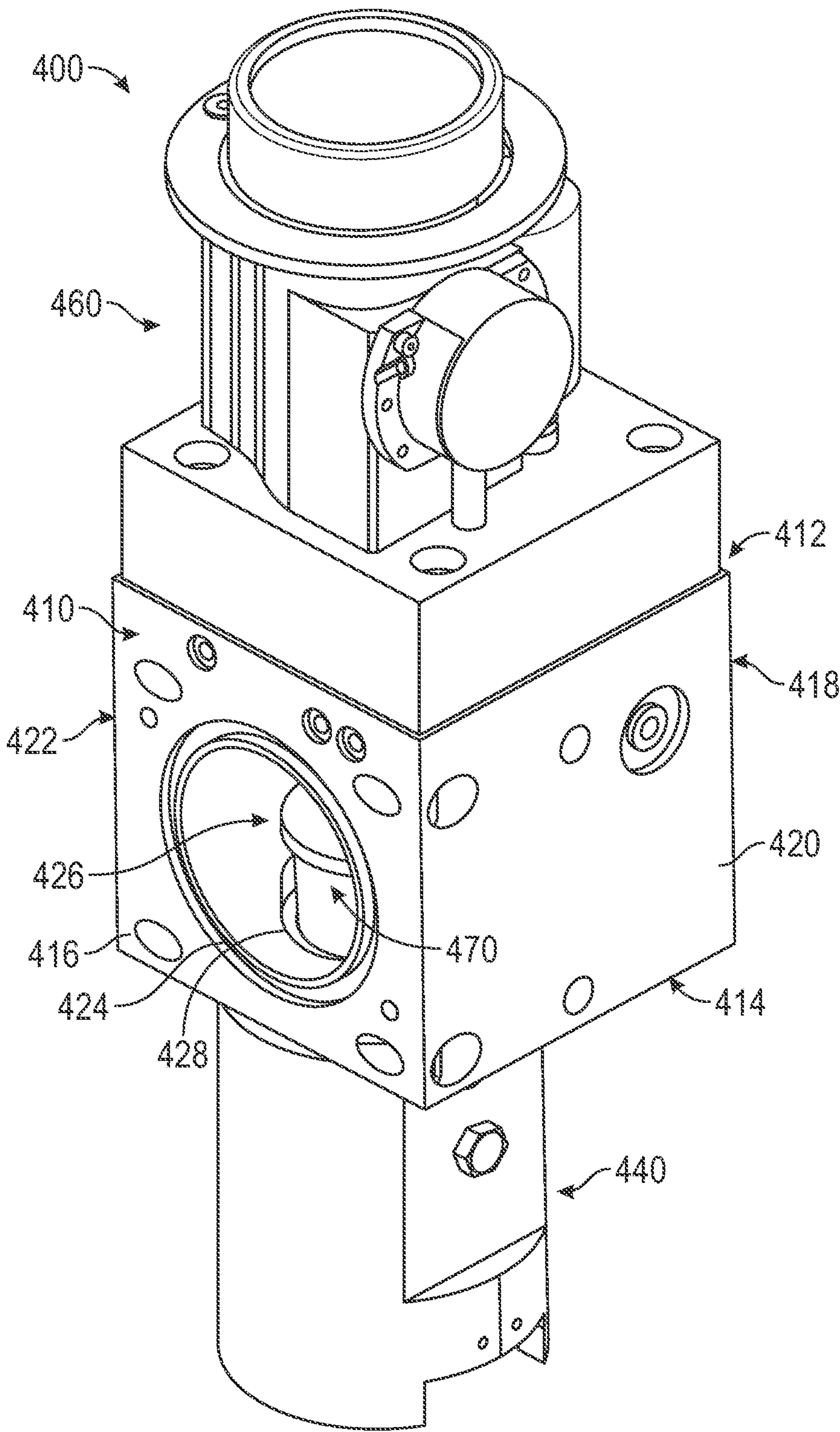


FIG. 8A

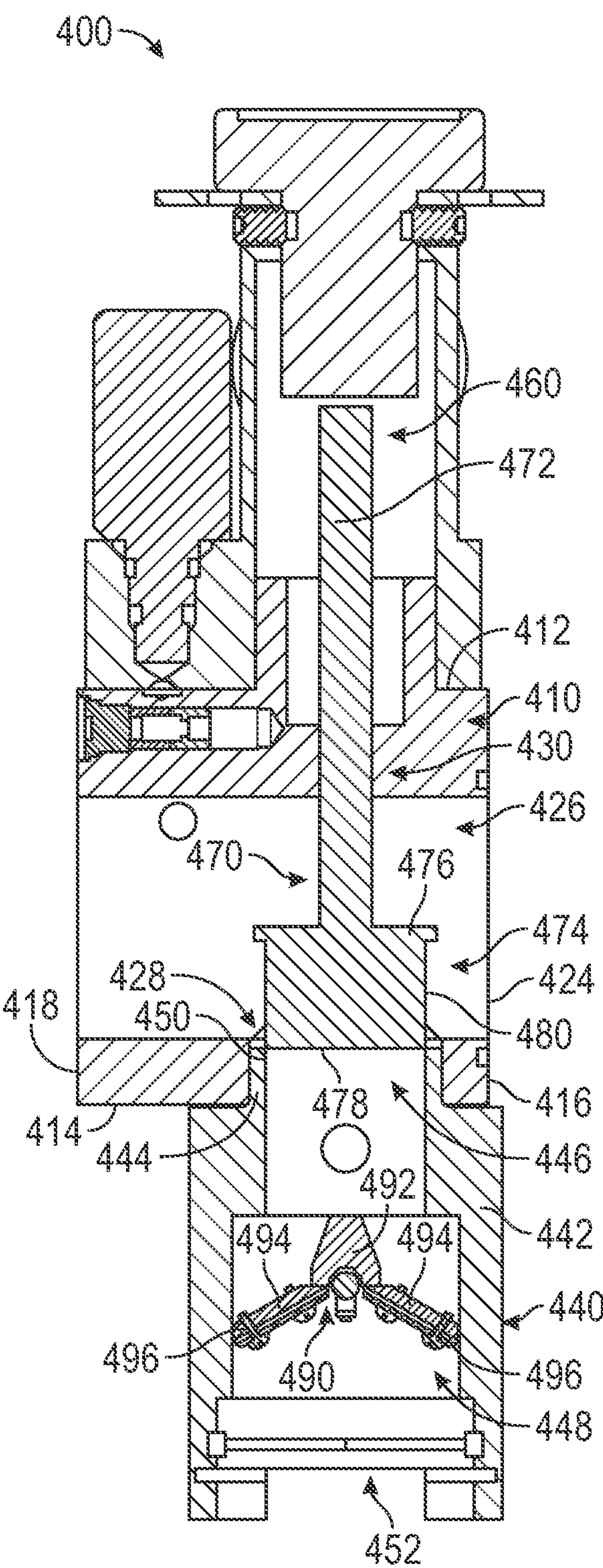


FIG. 8B

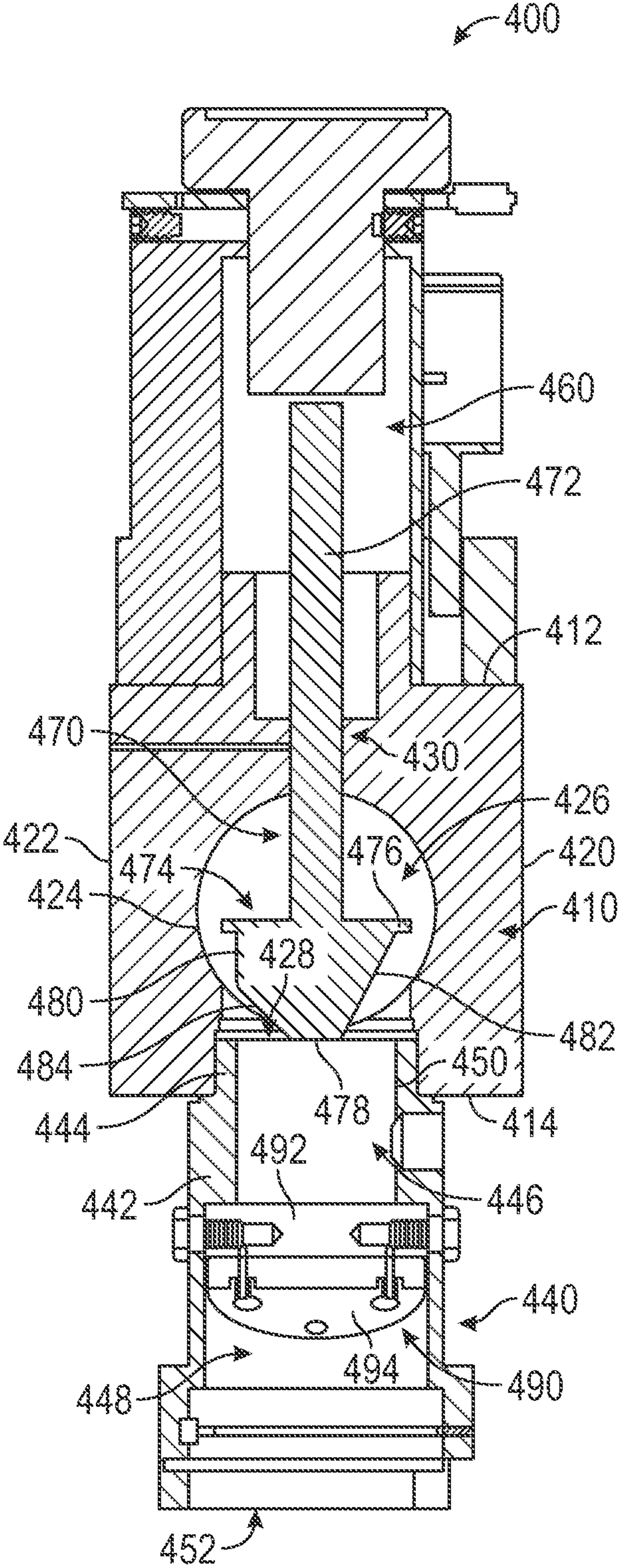


FIG. 8C

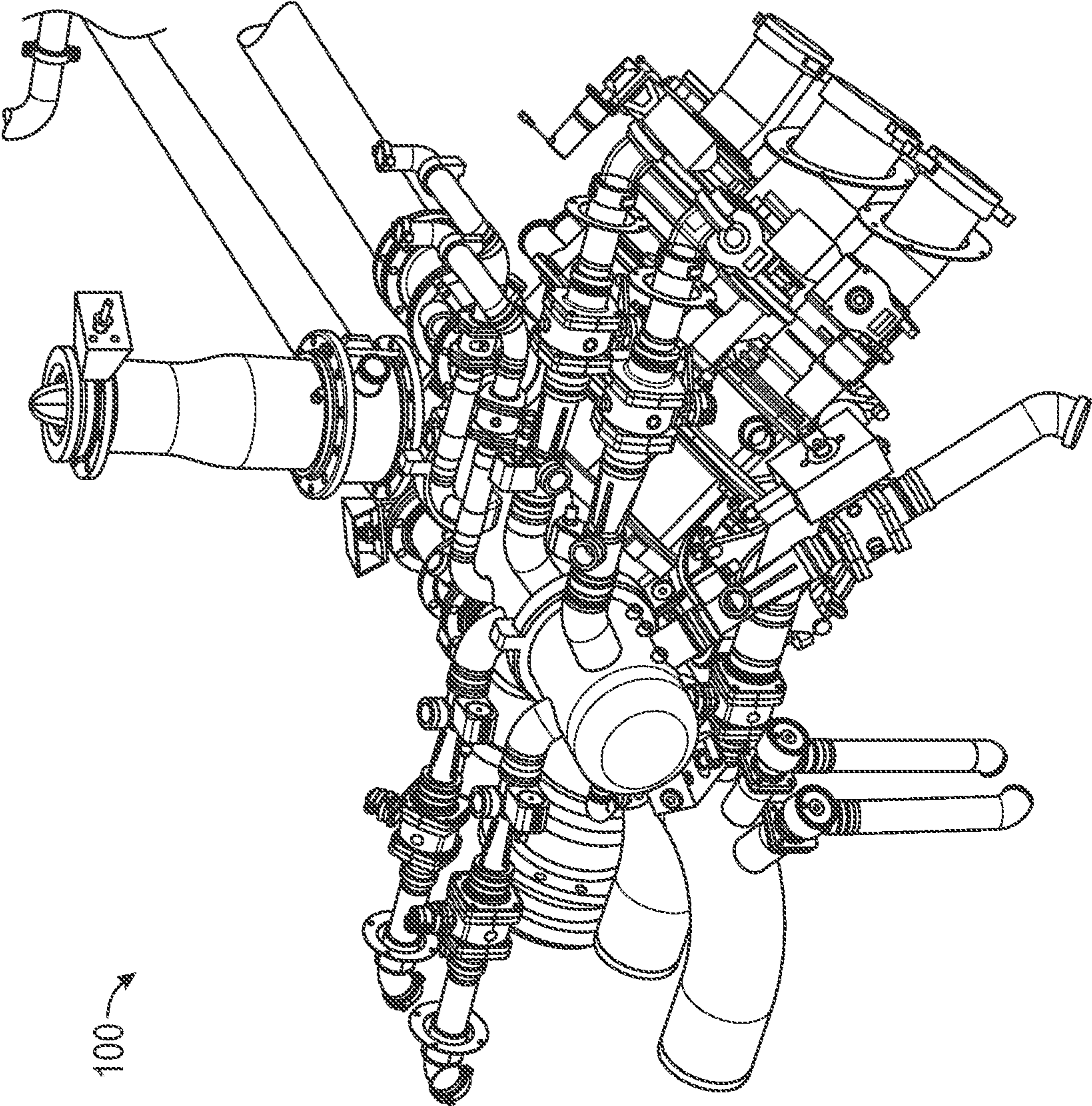


FIG. 9

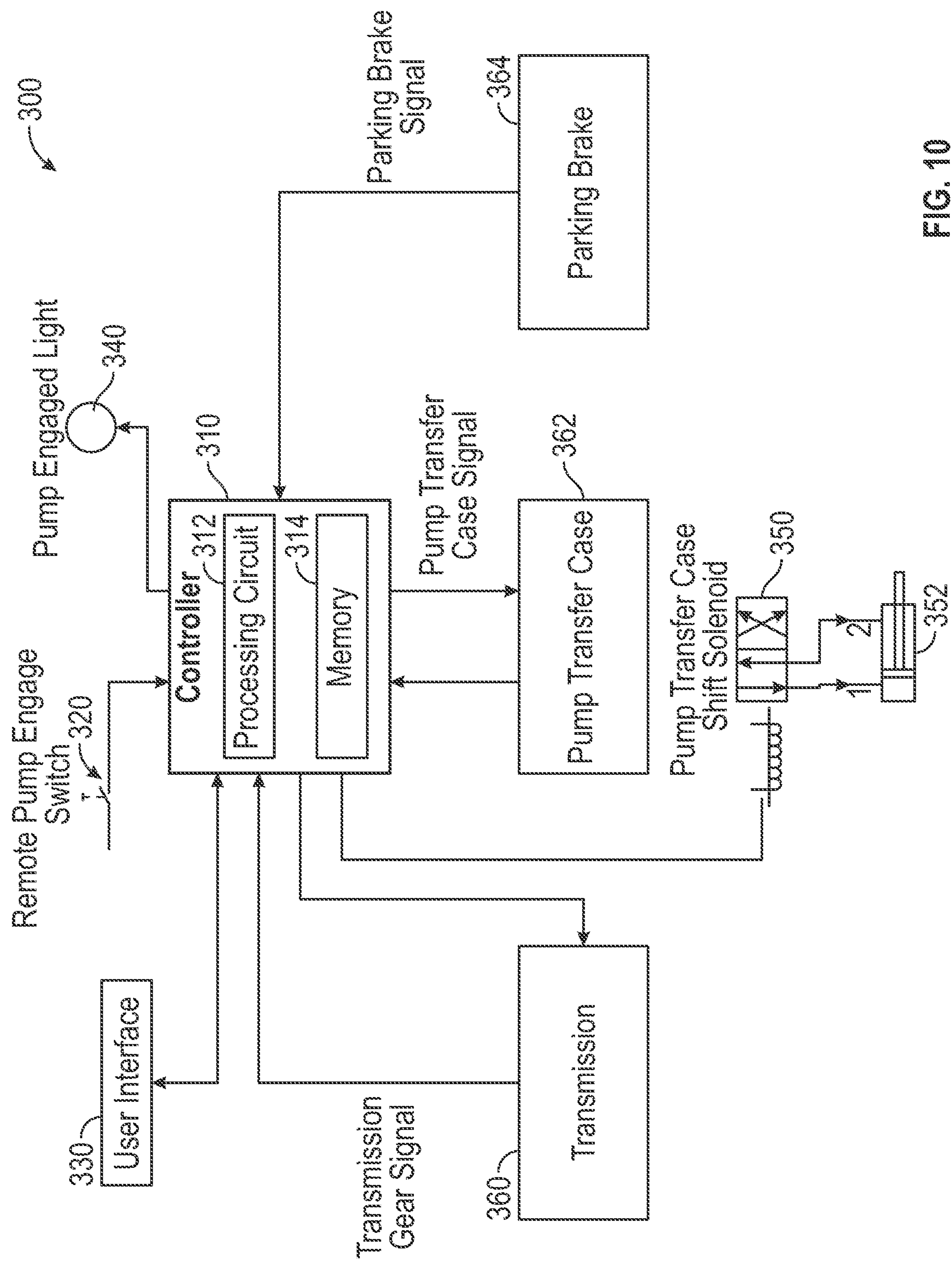


FIG. 10

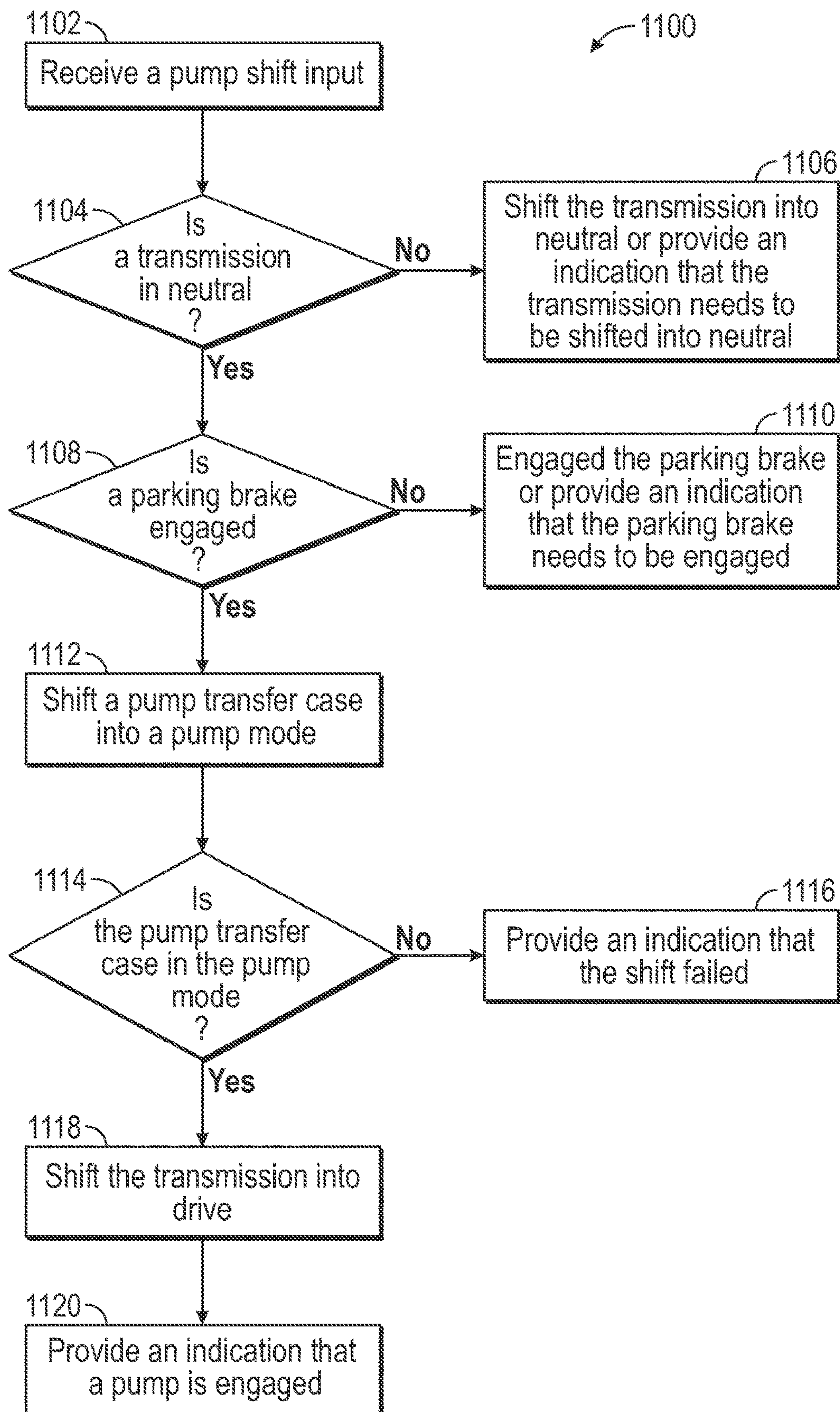


FIG. 11

1**FLUID DELIVERY SYSTEM FOR A FIRE APPARATUS****CROSS-REFERENCE TO RELATED PATENT APPLICATIONS**

This application is a continuation of U.S. patent application Ser. No. 17/531,017, filed Nov. 19, 2021, which is a continuation of U.S. patent application Ser. No. 16/035,264, filed Jul. 13, 2018, which claims the benefit of and priority to U.S. Provisional Patent Application No. 62/532,817, filed Jul. 14, 2017, all of which are incorporated herein by reference in their entireties.

BACKGROUND

Water and/or other agents (e.g., foam fire suppressants) may be transported by a fire apparatus to an emergency site to be discharged and facilitate extinguishment.

SUMMARY

One embodiment relates to a ratio controller for a fluid system of a fire apparatus. The ratio controller includes a housing defining a mixing chamber. The housing has a water inlet configured to receive water from a water supply, an agent inlet configured to receive agent from an agent supply, an outlet configured to output an agent-water solution, a first pressure port extending through a sidewall of the housing at a first position proximate the water inlet, and a second pressure port extending through the sidewall of the housing at a second position proximate the mixing chamber.

Another embodiment relates to a ratio controller. The ratio controller includes a housing having a sidewall. The housing defining a first inlet positioned at a first end of the housing and configured to receive a first fluid flow, a second inlet positioned along the sidewall and configured to receive a second fluid flow, an outlet positioned at an opposing second end of the housing and configured to output a mixture of the first fluid flow and the second fluid flow, a mixing chamber positioned between the first inlet and the outlet, a first port positioned along and extending through the sidewall proximate the first inlet, and a second port positioned along and extending through the sidewall proximate the mixing chamber.

Still another embodiment relates to a ratio controller. The ratio controller includes a housing. The housing define a mixing chamber, a first inlet positioned at a first end of the mixing chamber, an outlet positioned at an opposing second end of the mixing chamber, a second inlet positioned at a first location around a periphery of the mixing chamber between the first inlet and the outlet where the second inlet is configured to receive a second fluid input, and a third inlet positioned at a second location around the periphery of the mixing chamber between the first inlet and the outlet.

The invention is capable of other embodiments and of being carried out in various ways. Alternative exemplary embodiments relate to other features and combinations of features as may be recited herein.

BRIEF DESCRIPTION OF THE DRAWINGS

The disclosure will become more fully understood from the following detailed description, taken in conjunction with the accompanying figures, wherein like reference numerals refer to like elements, in which:

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FIG. 1 is a left side view of a fire fighting vehicle, according to an exemplary embodiment;

FIG. 2 is a left side view of a fire fighting vehicle, according to another exemplary embodiment;

FIG. 3 is a schematic diagram of a fluid delivery system for the fire fighting vehicles of FIGS. 1 and 2, according to an exemplary embodiment;

FIG. 4 is a schematic diagram of a fluid delivery system for the fire fighting vehicles of FIGS. 1 and 2, according to another exemplary embodiment;

FIGS. 5A-5D are various views of a ratio controller of the fluid delivery systems of FIGS. 3 and 4, according to an exemplary embodiment;

FIGS. 6A-6F are various views of a combined metering and shut-off valve assembly of the fluid delivery system of FIG. 4, according to an exemplary embodiment;

FIGS. 7A-7E are various views of a ball of the combined metering and shut-off valve assembly of the fluid delivery system of FIGS. 6A-6F, according to an exemplary embodiment;

FIGS. 8A-8C are various views of a combined metering and shut-off valve assembly of the fluid delivery system of FIG. 4, according to another exemplary embodiment;

FIG. 9 is a perspective view of a portion of the fluid delivery systems of FIGS. 3 and 4, according to an exemplary embodiment;

FIG. 10 is a schematic diagram of a pump engagement system for a pump of the fire fighting vehicles of FIGS. 1 and 2, according to an exemplary embodiment; and

FIG. 11 is a flow diagram of a method for a shifting a pump into a pump mode, according to an exemplary embodiment.

DETAILED DESCRIPTION

Before turning to the figures, which illustrate the exemplary embodiments in detail, it should be understood that the present application is not limited to the details or methodology set forth in the description or illustrated in the figures. It should also be understood that the terminology is for the purpose of description only and should not be regarded as limiting.

According to the exemplary embodiment shown in FIGS. 1 and 2, a vehicle (e.g., a fire apparatus, etc.), shown as fire fighting vehicle 10, includes a fluid supply system, shown as fluid delivery system 100. According to an exemplary embodiment, the fluid delivery system 100 is configured to provide (e.g., pump, etc.) a fluid (e.g., water, etc.) and/or an agent (e.g., foam, etc.) to aid in extinguishing a fire. According to the exemplary embodiment shown in FIG. 1, the fire fighting vehicle 10 is an aircraft rescue and firefighting ("ARFF") truck. According to the exemplary embodiment shown in FIG. 2, the fire fighting vehicle is a quint fire truck having an aerial ladder assembly 50. According to various alternative embodiments, the fire fighting vehicle 10 is a municipal fire fighting vehicle, a tiller fire apparatus, a forest fire apparatus, an aerial truck, a rescue truck, a tanker, or still another type of fire fighting vehicle or apparatus. According to still other embodiments, the vehicle is another type of vehicle (e.g., a military vehicle, a commercial vehicle, etc.).

As shown in FIGS. 1 and 2, the fire fighting vehicle 10 includes a chassis, shown as frame 12. The frame 12 supports a plurality of tractive elements, shown as front wheels 14 and rear wheels 16; a body assembly, shown as a rear section 18; and a cab, shown as front cabin 20. In one embodiment, the fire fighting vehicle 10 is a Striker® 6×6 with one front axle to support the front wheels 14 and two

rear axles to support the rear wheels **16** manufactured by Oshkosh Corporation®. In other embodiments, the fire fighting vehicle **10** is a Striker® 4×4, a Striker® 1500, a Striker® 3000, or a Striker® 4500 model manufactured by Oshkosh Corporation®. In still other embodiments, the fire fighting vehicle **10** is an Ascendant® model manufactured by Pierce Manufacturing®. Thus, the fire fighting vehicle **10** may include a different number of front axles and/or rear axles to support the front wheels **14** and the rear wheels **16** based on the application or model of the fire fighting vehicle **10**. In an alternative embodiment, the tractive elements are otherwise structured (e.g., tracks, etc.).

As shown in FIGS. **1** and **2**, the front cabin **20** is positioned forward of the rear section **18** (e.g., with respect to a forward direction of travel for the vehicle, etc.). According to an alternative embodiment, the front cabin **20** is positioned behind the rear section **18** (e.g., with respect to a forward direction of travel for the vehicle, etc.). According to an exemplary embodiment, the front cabin **20** includes a plurality of body panels coupled to a support (e.g., a structural frame assembly, etc.). The body panels may define a plurality of openings through which an operator accesses (e.g., for ingress, for egress, to retrieve components from within, etc.) an interior **24** of the front cabin **20**. As shown in FIGS. **1** and **2**, the front cabin **20** includes a pair of doors **22** positioned over the plurality of openings defined by the plurality of body panels. The doors **22** may provide access to the interior **24** of the front cabin **20** for a driver (or passengers) of the fire fighting vehicle **10**.

The front cabin **20** may include components arranged in various configurations. Such configurations may vary based on the particular application of the fire fighting vehicle **10**, customer requirements, or still other factors. The front cabin **20** may be configured to contain or otherwise support at least one of a number of occupants, storage units, equipment, and/or user interfaces. By way of example, the front cabin **20** may include a display, a joystick, buttons, switches, knobs, levers, touchscreens, a steering wheel, an accelerator pedal, a brake pedal, among other components. The user interface may provide the operator with control capabilities over the fire fighting vehicle **10** (e.g., direction of travel, speed, a transmission gear, etc.), one or more components of the fluid delivery system **100** (e.g., a turret, a pump, etc.), and still other components of the fire fighting vehicle **10** from within the front cabin **20**.

As shown in FIGS. **1** and **2**, the fire fighting vehicle **10** includes a powertrain, shown as powertrain **40**. The powertrain **40** of the fire fighting vehicle **10** may include a main driver (e.g., engine, motor, etc.), a transmission, a clutch, and/or a pump transfer case. The powertrain **40** may be coupled to a drivetrain (e.g., a drive shaft, a differential, an axle, etc. via the transmission, etc.) and/or a pump (e.g., a pump of the fluid delivery system **100** via the pump transfer case, etc.). According to an exemplary embodiment, the powertrain **40** (e.g., the engine, transmission, clutch, pump transfer case, etc. thereof) is coupled to and supported by the frame **12**. According to an exemplary embodiment, the engine receives fuel (e.g., gasoline, diesel, etc.) from a fuel tank and combusts the fuel to generate mechanical energy. The transmission receives the mechanical energy and provides an output to a drive shaft and/or the pump transfer case. The rotating drive shaft is received by a differential, which conveys the rotational energy of the drive shaft to a final drive or tractive element, such as the front wheels **14** and/or the rear wheels **16**. The front wheels **14** and/or the rear wheels **16** then propel or move the fire fighting vehicle **10**. The powertrain **40** may be configured to drive the front

wheels **14**, the rear wheels **16**, or a combination thereof (e.g., front-wheel-drive, rear-wheel-drive, all-wheel-drive, etc.). The driven pump transfer case may convey the mechanical energy provided by the transmission to a pump (e.g., a water pump, an agent pump, etc.) of the fluid delivery system **100** to drive a fluid (e.g., water, agent, etc.) through the fluid delivery system **100** to be used for fire suppression. According to an exemplary embodiment, the engine is a compression-ignition internal combustion engine that utilizes diesel fuel. In alternative embodiments, the engine is another type of driver (e.g., spark-ignition engine, fuel cell, electric motor, hybrid engine/motor, etc.) that is otherwise powered (e.g., with gasoline, compressed natural gas, hydrogen, electricity, etc.).

As shown in FIGS. **3** and **4**, the fluid delivery system **100** includes a first fluid circuit, shown as water circuit **110**; a second fluid circuit, shown as agent circuit **120**; a ratio controller, shown as ratio controller **140**; and a valve, shown as discharge valve **182**. As shown in FIGS. **3** and **4**, the water circuit **110** includes a first tank, shown as water tank **112**, and a first pump, shown as water pump **114**. In some embodiments, the water circuit **110** does not include the water tank **112**, but is configured to couple to an external water source (e.g., a fire hydrant, etc.). The water pump **114** is configured to pump water stored within the water tank **112** at a target flow rate (e.g., a target volumetric flow rate; 6000 gallons-per-minute (“gpm”), 3000 gpm, 1500 gpm, etc.; based on engine speed; based on a user input; etc.) through the water circuit **110** to the ratio controller **140**. According to an exemplary embodiment, the water pump **114** is coupled to and driven by the engine of the powertrain **40** via the pump transfer case thereof. In other embodiments, the water pump **114** is driven by a device designated solely for the water pump **114** (e.g., a motor, etc.).

As shown in FIG. **3**, the agent circuit **120** includes a second tank, shown as agent tank **122**; a second pump, shown as agent pump **124**; a metering device, shown as agent metering valve **126**; a blocking valve, shown as agent shut-off valve **130**; and a one-way valve, shown as agent check valve **132**. As shown in FIG. **4**, the agent circuit **120** does not include the agent metering valve **126** or the agent shut-off valve **130**, but rather the agent metering valve **126** and the agent shut-off valve **130** are replaced with a first single valve component, shown as combined agent metering and shut-off valve assembly **200**, or a second single valve component, shown as combined agent metering and shut-off valve assembly **400**. In some embodiments, the agent circuit **120** does not include the agent check valve **132** (e.g., in embodiments where the agent circuit **120** may include the combined agent metering and shut-off valve assembly **400** which may include an integrated check valve, etc.). The agent pump **124** is configured to drive agent stored within the agent tank **122** (e.g., at a target volumetric flow rate, X gallons-per-minute (“gpm”), based on the flow rate of the water entering the ratio controller **140**, based on a user input, etc.) through the agent circuit **120** to the ratio controller **140**. In some embodiments, the agent pump **124** is coupled to and driven by the engine of the powertrain **40** (e.g., via a power-take-off (“PTO”), etc.). In some embodiment, the agent pump **124** is driven by a device designated solely for the agent pump **124** (e.g., a motor, etc.).

As shown in FIGS. **1** and **2**, the water tank **112** and the agent tank **122** are disposed within the rear section **18** of the fire fighting vehicle **10**. In other embodiments, the water tank **112** and/or the agent tank **122** are otherwise positioned (e.g., disposed along a rear, front, roof, side, etc. of the fire fighting vehicle **10**). According to an exemplary embodi-

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ment, the water tank **112** and/or the agent tank **122** are corrosion and UV resistant polypropylene tanks. In other embodiments, the water tank **112** and/or the agent tank **122** are manufactured from another suitable material.

According to an exemplary embodiment, the water tank **112** is configured to store a fluid, such as water or another liquid. In one embodiment, the water tank **112** is a 3,000 gallon capacity tank. In another embodiment, the water tank **112** is a 1,500 gallon capacity tank. In still another embodiment, the water tank **112** is a 4,500 gallon capacity tank. In other embodiments, the water tank **112** has another capacity. In some embodiments, multiple water tanks **112** are disposed within and/or along the rear section **18** of the fire fighting vehicle **10**.

According to an exemplary embodiment, the agent tank **122** is configured to store an agent, such as a foam fire suppressant. According to an exemplary embodiment, the agent is an aqueous film forming foam ("AFFF"). AFFF is water-based and frequently includes hydrocarbon-based surfactant (e.g., sodium alkyl sulfate, etc.) and a fluorosurfactant (e.g., fluorotelomers, perfluorooctanoic acid, perfluorooctanesulfonic acid, etc.). AFFF has a low viscosity and spreads rapidly across the surface of hydrocarbon fuel fires. An aqueous film forms beneath the foam on the fuel surface that cools burning fuel and prevents evaporation of flammable vapors and re-ignition of fuel once it has been extinguished. The film also has a self-healing capability whereby holes in the film layer are rapidly resealed. In alternative embodiments, another agent is stored with the agent tank **122** (e.g., low-expansion foams, medium-expansion foams, high-expansion foams, alcohol-resistant foams, synthetic foams, protein-based foams, foams to be developed, etc.). In one embodiment, the agent tank **122** is a 420 gallon capacity tank. In another embodiment, the agent tank **122** is a 210 gallon capacity tank. In still another embodiment, the agent tank **122** is a 630 gallon capacity tank. In other embodiments, the agent tank **122** has another capacity. In some embodiments, multiple agent tanks **122** are disposed within and/or along the rear section **18** of the fire fighting vehicle **10**. The capacity of the water tank **112** and/or the agent tank **122** may be specified by a customer. It should be understood that water tank **112** and the agent tank **122** configurations are highly customizable, and the scope of the present application is not limited to particular size or configuration of the water tank **112** and the agent tank **122**.

As shown in FIGS. **3** and **4**, the fluid delivery system **100** optionally includes a first sensor, shown as water circuit sensor **102**, and a second sensor, shown as agent circuit sensor **104**. The water circuit sensor **102** may include one or more sensors variously positioned along the water circuit **110**. By way of example, the water circuit sensor(s) **102** may be positioned downstream of the water tank **112** and upstream of the water pump **114** and/or downstream of the water pump **114**. The water circuit sensor(s) **102** may include (i) one or more water pressure sensors positioned to facilitate monitoring the pressure of the water within water circuit **110** upstream and/or downstream of the water pump **114** and/or (ii) a water flow meter positioned to facilitate monitoring the flow rate (e.g., volumetric flow rate, etc.) of the water flowing through the water circuit **110** to the ratio controller **140**.

The agent circuit sensor **104** may include one or more sensors variously positioned along the agent circuit **120**. By way of example, the agent circuit sensor(s) **104** may be positioned downstream of the agent tank **122** and upstream of the agent pump **124**, downstream of the agent pump **124**

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and upstream of the agent metering valve **126**, downstream of the agent metering valve **126** and upstream of the agent shut-off valve **130**, downstream of the agent shut-off valve **130** and upstream of the agent check valve **132**, downstream of the agent check valve **132**, downstream of the agent pump **124** and upstream of the combined agent metering and shut-off valve assembly **200**, downstream of the combined agent metering and shut-off valve assembly **200** and the agent check valve **132**, downstream of the agent pump **124** and upstream of the combined agent metering and shut-off valve assembly **400**, and/or downstream of the combined agent metering and shut-off valve assembly **400**. The agent circuit sensor(s) **104** may include (i) one or more agent pressure sensors positioned to facilitate monitoring the pressure of the agent at any desired location within the agent circuit **120** and/or (ii) an agent flow meter positioned to facilitate monitoring the flow rate (e.g., volumetric flow rate, etc.) of the agent flowing through the agent circuit **120** to the ratio controller **140**.

As shown in FIGS. **3** and **4**, the agent metering valve **126**, the combined agent metering and shut-off valve assembly **200**, and/or the combined agent metering and shut-off valve assembly **400** are optionally coupled to a controller, shown as valve controller **128**. The agent metering valve **126**, the combined agent metering and shut-off valve assembly **200**, and/or the combined agent metering and shut-off valve assembly **400** may thereby be configured as a non-self-adjusting or non-continuous metering valve (e.g., manually/mechanically set and controlled, in embodiments where the fluid delivery system **100** does not include the valve controller **128**, etc.) and/or a self-adjusting, continuous metering valve (e.g., automatically/electronically controlled, in embodiments where the fluid delivery system **100** includes the valve controller **128**, etc.).

According to an exemplary embodiment, the agent metering valve **126** is configured to selectively restrict the amount of agent flowing therethrough such that the agent mixes with the water (e.g., within the ratio controller **140**, etc.) to create an agent-water solution with an appropriate agent-to-water ratio. In embodiments where the fluid delivery system **100** does not include the valve controller **128**, the agent metering valve **126** may be any type of metering valve (e.g., a ball valve, a spool valve, a v-notch valve, etc.) that does not provide self-adjustment over a continuous range of agent-to-water ratios. By way of example, the agent metering valve **126** may have multiple predefined orifices and/or valve settings that provide discrete adjustment of the agent-to-water ratio of the agent-water solution in specific, predefined increments (e.g., 0.5%, 1%, 3%, 6%, etc., etc.).

In embodiments where the fluid delivery system **100** includes the valve controller **128**, the agent metering valve **126** may be a self-adjusting, adaptive metering valve configured to provide a continuous range of agent-to-water ratios (e.g., any agent-to-water ratio between 0% and 10%, etc.) for all rated water flows of the fluid delivery system **100**. By way of example, the valve controller **128** may be configured to receive an indication of the water flow rate entering the ratio controller **140**. The indication of the water flow rate may be provided by a signal from the water circuit sensor **102** (e.g., a water flow meter, etc.) and/or a signal from the ratio controller **140** (e.g., a flow meter of the ratio controller **140**, etc.). The valve controller **128** may be further configured to receive an indication of a desired agent-to-water ratio for the agent-water solution (e.g., from an operator using a user interface of the fire fighting vehicle **10**, etc.). The valve controller **128** may be configured to (i) receive the indication of the water flow rate and the indica-

tion of the desired agent-to-water ratio and (ii) adaptively adjust (e.g., modulate, vary, etc.) an orifice size or valve position of the agent metering valve **126** as the water flow rate fluctuates (e.g., the orifice size or valve position is increased as the water flow rate increases such that more agent is provided, the orifice size or valve position is decreased as the water flow rate decreases such that less agent is provided, etc.) to maintain an accurate agent concentration within the agent-water solution. According to an exemplary embodiment, such a self-adjusting agent metering valve **126** is configured to facilitate providing agent-water solutions having an agent-to-water ratio within 0.1% accuracy of the desired agent-to-water ratio, while traditional agent metering valves may facilitate providing agent-water solutions having agent-to-water ratios within 1% accuracy. Therefore, at a water flow rate of 6000 gpm, a traditional agent metering valve may provide up to 60 gallons per minute of excess agent, while the self-adjusting agent metering valve may provide less than 6 gallons per minute of potential excess agent.

The valve controller **128** may be configured to determine the orifice size or valve position at which to adjust the agent metering valve **126** by storing a few calibration points for various agent-to-water ratios. By way of example, the valve controller **128** may be configured to store a few (e.g., two, three, four, five, etc.) predetermined orifice sizes or valve positions for a few (e.g., two, three, four, five, etc.) predetermined water flow rates (e.g., 1500 gpm, 3000 gpm, 4500 gpm, 6000 gpm, etc.) that provide specific agent-to-water ratios (e.g., common agent-to-water ratios such as 0.3%, 0.5%, 1%, 3%, 6%, etc.). For example, the valve controller **128** may store three water flow rates and three corresponding orifice sizes or valve positions that provide each specific agent-to-water ratio. From such predefined parameters, a curve may be generated by the valve controller **128** for each of the predefined specific agent-to-water ratios (e.g., based on the predefined orifice sizes and water flow rates for each agent-to-water ratios, etc.). Therefore, if an operator selects one of the predefined agent-to-water ratios (e.g., 0.3%, 1%, 3%, 6%, etc.), the orifice size or position of the agent metering valve may be determined by the valve controller **128** at the point at which the current water flow rate intersect the curve for the selected, predefined agent-to-water ratio. However, if an operator selects an agent-to-water ratio that is not predefined (e.g., a ratio other than 0.3%, 0.5%, 1%, 3%, 6%, etc.), the valve controller **128** may be configured to derive the orifice size or position of the agent metering valve **126**. By way of example, if an agent-to-water ratio of 0.75% is selected, the predefined orifice sizes or positions of the agent metering valve **126** from the upper agent-to-water ratio curve (e.g., 1% curve, etc.) and the lower agent-to-water ratio curve (e.g., the 0.5% curve, etc.) may be averaged for each predetermined water flow rate (e.g., 1500 gpm, 3000 gpm, 4500 gpm, 6000 gpm, etc.) to generate an intermediate curve for the selected agent-to-water ratio (e.g., 0.75%, etc.). The valve controller **128** may then determine the orifice size or position of the agent metering valve **126** at the point where the current water flow rate intersect the derived curve.

According to an exemplary embodiment, the agent shut-off valve **130** is configured to facilitate selectively isolating the agent circuit **120** from the ratio controller **140**. By way of example, the agent shut-off valve **130** may (i) prevent agent from passing therethrough and reaching the ratio controller **140** when arranged in a first configuration (e.g., a closed configuration, etc.) such that only water is discharged from the fluid delivery system **100** and (ii) allow agent to

pass freely therethrough and mix with the water within the ratio controller **140** when arranged in a second configuration (e.g., an open configuration, etc.) such that an agent-water solution is discharged from the fluid delivery system **100**.

The agent shut-off valve **130** may be a manually-actuated valve or an electronically-actuated valve.

According to an exemplary embodiment, the combined agent metering and shut-off valve assembly **200** and/or the combined agent metering and shut-off valve assembly **400** are configured to replace and perform the various function described herein in relation to the agent metering valve **126**, the agent shut-off valve **130**, and/or the agent check valve **132**.

As a brief overview of the combined agent metering and shut-off valve assembly **200**, the agent metering and shut-off valve assembly **200** includes a ball that defines an elongated “V” notch that variably restricts agent flow through the combined agent metering and shut-off valve assembly **200**. The combined agent metering and shut-off valve assembly **200** has an inlet, an outlet, and a 90 degree flow path extending therebetween. The ball is capable of shutting the “V” notch completely (e.g., thereby functioning as both the agent metering valve **126** and the agent shut-off valve **130**, etc.). By lengthening the “V” notch, agent flow can be accurately controlled over a greater range of agent and water flow rates.

As shown in FIGS. 6A-6F, the combined agent metering and shut-off valve assembly **200** includes a housing, shown as valve body **210**; an inner sleeve, shown as flow directing conduit **240**; an adjuster, shown as ball adjuster **250**; an extension, shown as valve spout **260**; a plate, shown as end plate **268**; and a flow restrictor, shown as ball **270**. As shown in FIGS. 6A-6F, the valve body **210** has a first end, shown as bottom end **212**; an opposing second end, shown as top end **214**; a first lateral face, shown as front face **216**; and an opposing second face, shown as rear face **218**. As shown in FIG. 6F, the bottom end **212** of the valve body **210** defines an aperture, shown as valve body inlet **220**. The top end **214** of the valve body **210** defines a passage, shown as top passage **222**. The rear face **218** of the valve body **210** defines an aperture, shown as rear opening **224**. The front face **216** of the valve body **210** defines an opening, shown as valve body outlet **226**. The valve body inlet **220**, the top passage **222**, the rear opening **224**, and the valve body outlet **226** each lead into an internal cavity, shown as interior chamber **228**, defined by the valve body **210**.

As shown in FIG. 6F, the flow directing conduit **240** is received by the valve body inlet **220** and at least partially disposed within the interior chamber **228** of the valve body **210**. The flow directing conduit **240** includes an inlet, shown as agent inlet **242**, positioned at the valve body inlet **220** at the bottom end **212** of the valve body **210** and an outlet, shown as agent outlet **244**, positioned to align with the valve body outlet **226** at the front face **216** of the valve body **210**. According to an exemplary embodiment, the agent outlet **244** is positioned perpendicularly relative to the agent inlet **242** such that the flow directing conduit **240** directs incoming agent along a ninety degree flow path (e.g., the agent comes in the bottom end **212** and exits the front face **216**, etc.). As shown in FIG. 6F, a top end of a sidewall of the flow directing conduit **240** defines an aperture, shown as aperture **246**.

As shown in FIGS. 6A and 6F, the ball adjuster **250** is received by the top passage **222** of the valve body **210**. As shown in FIGS. 6A and 6C-6F, the ball adjuster **250** includes a handle, shown as knob **252**. The knob **252** may be manually actuated by an operator such that the ball adjuster

250 is rotated within the interior chamber **228** of the valve body **210**. In some embodiments, the ball adjuster **250** is electrically actuated (e.g., with an electric actuator, a solenoid, etc.) by the valve controller **128** (e.g., such that the combined agent metering and shut-off valve assembly **200** is self-adjusting, an adaptive metering valve, etc.). As shown in FIG. 6F, the ball adjuster **250** includes an interface, shown as ball key **254**, having a first projection, shown as first cylindrical protrusion **256**, extending therefrom. The first cylindrical protrusion **256** has a second projection, shown as second cylindrical protrusion **258**, extending therefrom and received by the aperture **246** of the flow directing conduit **240**.

As shown in FIGS. 6A-6D and 6F, the valve spout **260** includes a coupler, shown as flange **262**, with a protrusion, shown as outlet conduit **264**, extending therefrom. As shown in FIGS. 6A, 6C, and 6F, the flange **262** and the outlet conduit **264** cooperatively define a passage, shown as discharge passage **266**. As shown in FIGS. 6A-6D and 6F, the flange **262** is coupled to the valve spout **260** to the front face **216** of the valve body **210** such that the discharge passage **266** aligns with the valve body outlet **226** to receive agent therefrom. As shown in FIG. 6F, a resilient member, shown as seal **230**, is positioned between the flange **262** and the front face **216** of the valve body **210** to prevent agent from seeping through the interface therebetween. As shown in FIGS. 6A, 6B, and 6D-6F, the end plate **268** is coupled to the rear face **218** of the valve body **210**. The end plate **268** is positioned to enclose the rear opening **224** in the rear face **218** of the valve body **210**.

As shown in FIG. 6F, the ball **270** is disposed within the interior chamber **228** of the valve body **210**. As shown in FIGS. 6F-7E, the ball **270** has an outer wall, shown as shell **272**, having a first end, shown as top end **290**, and an opposing second end, shown as bottom end **292**. According to an exemplary embodiment, the shell **272** is substantially spherical. According to the exemplary embodiment shown in FIGS. 7C and 7D, the bottom end **292** of the shell **272** has a flat surface. In other embodiments, the bottom end **292** of the shell **272** is spherical. According to the exemplary embodiment shown in FIGS. 7A and 7E, the shell **272** has a partially lobed or camed profile.

As shown in FIGS. 6F-7C, the top end **290** of the shell **272** of the ball **270** defines a cutout, shown as keyed recess **274**, and an aperture, shown as through-hole **276**. As shown in FIG. 6F, the keyed recess **274** receives the ball key **254** of the ball adjuster **250** and the through-hole **276** receives the first cylindrical protrusion **256**. According to an exemplary embodiment, the engagement between the keyed recess **274** and the ball key **254** facilitates rotating the ball **270** within the interior chamber **228** with the ball adjuster **250**. According to an exemplary embodiment, the ball **270** is rotatable through two hundred degrees of rotation. Rotating the ball **270** two hundred degrees may facilitate completely shutting off the flow of agent through the valve body **210** (e.g., the ball **270** functions similar to the agent shut-off valve **130**, etc.). In other embodiments, the ball **270** is rotatable more than or less than two hundred degrees (e.g., 90 degrees, 180 degrees, 225 degrees, 270 degrees, 315 degrees, 360 degrees, anywhere therebetween, etc.).

As shown in FIGS. 6F, 7B, and 7E, the bottom end **292** of the shell **272** defines an aperture, shown as ball inlet **277**, that leads to an interior cavity, shown as ball chamber **278**, of the ball **270**. As shown in FIG. 6F, the ball inlet **277** receives the flow directing conduit **240** such that the agent outlet **244** of the flow directing conduit **240** is disposed within the ball chamber **278** of the ball **270**.

As shown in FIGS. 6F and 7B-7D, the shell **272** of the ball **270** defines a cutout or notch, shown as variable flow outlet **280**, extending at least partially around the periphery of the shell **272** (e.g., 60 degrees, 90 degrees, 180 degrees, 225 degrees, 270 degrees, 315 degrees, 330 degrees, anywhere therebetween, etc.). The variable flow outlet **280** has a first end, shown as minimum end **282**; a second end, shown as maximum end **284**; and a linearly angled profile, shown as “V” profile **286**, extending between the minimum end **282** and the maximum end **284**. In other embodiments, the variable flow outlet **280** has a non-linear profile (e.g., parabolic, stepped, etc.). According to an exemplary embodiment, the ball **270** is rotatable within the interior chamber **228** of the valve body **210** such that the position of the agent outlet **244** of the flow directing conduit **240** along the “V” profile **286** of the variable flow outlet **280** may be selectively varied (e.g., between the minimum end **282** and the maximum end **284**, etc.). By way of example, the ball **270** may be rotated into a first position such that the variable flow outlet **280** is in a position that effectively seals the agent outlet **244** of the flow directing conduit **240**. By way of another example, the ball **270** may be rotated into a second position such that the minimum end **282** of the variable flow outlet **280** aligns with the agent outlet **244** of the flow directing conduit **240**, effectively setting the amount of agent that flows through the valve body **210** and out of the valve spout **260** at the minimum agent flow rate. By way of yet another example, the ball **270** may be rotated into a third position such that the maximum end **284** of the variable flow outlet **280** aligns with the agent outlet **244** of the flow directing conduit **240**, effectively setting the amount of agent that flows through the valve body **210** and out of the valve spout **260** at the maximum agent flow rate. The ball **270** may further be rotated into a position between the second position and the third position to set the amount of agent that flows through the valve body **210** and out of the valve spout **260** somewhere between the minimum agent flow rate and the maximum agent flow rate (e.g., to provide the required amount of agent to the ratio controller **140** such that the agent-water solution has the appropriate agent-to-water ratio, etc.).

As a brief overview of the combined agent metering and shut-off valve assembly **400**, the agent metering and shut-off valve assembly **400** includes a plunger that includes a portion that defines a non-uniform “V-shaped” profile that variably restricts agent flow through the combined agent metering and shut-off valve assembly **400**. The combined agent metering and shut-off valve assembly **400** has an inlet, an outlet, and a 90 degree flow path extending therebetween. The plunger is capable of isolating or blocking the non-uniform “V-shaped” profile completely (e.g., thereby functioning as both the agent metering valve **126** and the agent shut-off valve **130**, etc.). By providing a non-uniform “V-shaped” profile, agent flow can be accurately controlled over a greater range of agent and water flow rates. In some embodiments, the combined agent metering and shut-off valve assembly **400** also includes an integrated check valve (e.g., thereby functioning as all three of the agent metering valve **126**, the agent shut-off valve **130**, and the agent check valve **132**, etc.).

As shown in FIGS. 8A-8C, the combined agent metering and shut-off valve assembly **400** includes a housing, shown as valve body **410**; an extension, shown as valve spout **440**; a driver (e.g., a solenoid, an electric actuator, a manual actuator, etc.), shown as actuator **460**; a flow restrictor or plunger, shown as needle **470**; and a one-way valve, shown as integrated check valve **490**. In some embodiments, the

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integrated check valve 490 eliminates the need for the agent check valve 132 along the agent circuit 120. In some embodiments, the combined agent metering and shut-off valve assembly 400 does not include the integrated check valve 490.

As shown in FIGS. 8A-8C, the valve body 410 is a rectangular prism having a first end, shown as top end 412; an opposing second end, shown as bottom end 414; a first face, shown as left face 416; a second face, shown as right face 418; a third face, shown as front face 420; and a fourth face, shown as rear face 422. In other embodiments, the valve body 410 has another shape (e.g., a cylinder, a cube, etc.). As shown in FIGS. 8A-8C, the left face 416 defines a first aperture, shown as valve body inlet 424, the bottom end 414 defines a second aperture, shown as valve body outlet 428, and the top end 412 defines a third aperture, shown as rod aperture 430. The valve body 410 defines a first chamber, shown as inlet chamber 426, that connects the valve body inlet 424 to the valve body outlet 428.

As shown in FIGS. 8B and 8C, the valve spout 440 has a first portion, shown as body 442, and a second portion, shown as flange 444, extending from a first end of the body 442 and having a diameter less than a diameter of the body 442. An opposing second end of the body 442 defines an outlet, shown as valve spout outlet 452, and the flange 444 defines an inlet, shown as valve spout inlet 450. The valve spout 440 defines a second, intermediate chamber, shown as intermediate chamber 446, and a third chamber, shown as outlet chamber 448. The intermediate chamber 446 and the outlet chamber 448 connect the valve spout inlet 450 and the valve spout outlet 452.

As shown in FIGS. 8A-8C, the valve spout 440 extends from the bottom end 414 of the valve body 410. As shown in FIGS. 8B and 8C, the flange 444 interfaces with and is received by the valve body outlet 428 such that the valve spout inlet 450 aligns with the valve body outlet 428, connecting the inlet chamber 426 to the intermediate chamber 446. In some embodiments, the valve body 410 and the valve spout 440 are integrally formed (e.g., a single, unitary structure, etc.). As shown in FIGS. 8B and 8C, the inlet chamber 426, the intermediate chamber 446, and the outlet chamber 448 cooperatively form a flow path from the valve body inlet 424 to the valve spout outlet 452. According to an exemplary embodiment shown in FIGS. 8B and 8C, the valve spout outlet 452 is positioned perpendicularly relative to the valve body inlet 424 such that incoming agent to the valve body 410 flows along a ninety degree flow path (e.g., the agent comes into the inlet chamber 426 through the valve body inlet 424 in the left face 416 of the valve body 410, exits the bottom end 414 of the valve body 410 through the valve body outlet 428 and the valve spout inlet 450 into the intermediate chamber 446, then through the outlet chamber 448 to the valve spout outlet 452, etc.).

As shown in FIGS. 8B and 8C, the needle 470 includes a shaft, shown as rod 472, having a first end coupled to the actuator 460 and an opposing second end that extends through the rod aperture 430 into the inlet chamber 426 of the valve body 410 and has a head (e.g., a plunger head, etc.), shown as variable flow head 474, coupled thereto. According to an exemplary embodiment, the actuator 460 is positioned and configured to variably reposition the needle 470 between a first, fully-extended position and a second, fully-retracted position (e.g., based on inputs received from the valve controller 128, etc.). In some embodiments, the actuator 460 is electronically controlled by the valve controller 128. In some embodiments, the actuator 460 is additionally or alternatively manually operable. By way of

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example, selectively repositioning the variable flow head 474 into the first, fully-extended position may position the variable flow head 474 such that the inlet chamber 426 is effectively sealed from the intermediate chamber 446 and the outlet chamber 448 to prevent any agent flow therebetween. By way of another example, selectively repositioning the variable flow head 474 into the second, fully-retracted position may position the variable flow head 474 such that agent flow from the inlet chamber 426 to the intermediate chamber 446 and the outlet chamber 448 is substantially uninhibited.

As shown in FIGS. 8B and 8C, the variable flow head 474 include a top portion, shown as annular ring 476, coupled to the opposing second end of the rod 472; a bottom portion, shown as bottom 478; and a sidewall (e.g., a cylindrical sidewall, etc.), shown as peripheral wall 480, extending between the annular ring 476 and the bottom 478 of the variable flow head 474 and having a diameter less than that of the annular ring 476. According to an exemplary embodiment, the annular ring 476 has a diameter that is larger than the diameter of the valve spout inlet 450 but that is less than or substantially equal to the diameter of the valve body outlet 428. The annular ring 476 may therefore be received by the valve body outlet 428 and engage with the end of the flange 444 of the valve spout 440 when the needle 470 is selectively repositioned into the first, fully-extended position and, thereby, selectively seal the inlet chamber 426 from the intermediate chamber 446 and the outlet chamber 448, restricting agent flow therebetween.

As shown in FIG. 8C, a portion of the peripheral wall 480 (e.g., a notched portion, a portion that is cutout from the peripheral wall 480 of the variable flow head 474, etc.) defines a non-uniform "V-shaped" profile having a first portion, shown first angled wall 482, and an opposing second portion, shown as second angled wall 484. The first angled wall 482 extends linearly at a first angle from the annular ring 476 along a first side of the peripheral wall 480 to the bottom 478 toward the center of the variable flow head 474, while the second angled wall 484 extends linearly at a second, different angle from a position along an opposing second side of the peripheral wall 480 between the annular ring 476 and the bottom 478 (e.g., approximately half way down the peripheral wall 480, etc.) to the bottom 478 toward the center of the variable flow head 474. According to an exemplary embodiment, the first angle is less than the second angle (e.g., the first angled wall 482 is less steep or has a lesser slope than the second angled wall 484, etc.). In other embodiments, the first angled wall 482 and/or the second angled wall 484 extend at different angles and/or from other positions along the peripheral wall 480. In some embodiments, the first angled wall 482 and/or the second angled wall 484 have a non-linear profile (e.g., curved, parabolic, etc.).

According to an exemplary embodiment, the variable flow head 474 is configured to facilitate providing fine and precise control of agent flow through the combined agent metering and shut-off valve assembly 400 in a first sub-set of positions for lower agent percentages of the agent-water solution (e.g., between the first, fully extended position and an intermediate position, etc.) and provide greater agent flow through the combined agent metering and shut-off valve assembly 400 in a second sub-set of positions for high agent percentages of the agent-water solution (e.g., between the intermediate position and the second, fully-retracted position, etc.). By way of example, while the variable flow head 474 is at least partially extended through the valve body outlet 428 and the valve spout inlet 450 (e.g., between the

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first, fully extended position and the intermediate position, etc.) such that the peripheral wall **480** adjacent the second angled wall **484** is in contact with the interior wall of the intermediate chamber **446**, isolating the second angled wall **484** from the inlet chamber **426**, agent may only flow through one side of the non-uniform “V-shaped” profile (i.e., through a first gap formed between the first angled wall **482** and the interior wall of the intermediate chamber **446**). As the variable flow head **474** is retracted from the intermediate chamber **446**, the first gap formed between the first angled wall **482** and the interior wall of the intermediate chamber **446** continues to increase in size, and as a result the agent flow therethrough increases. However, once the intermediate position is reached, the peripheral wall **480** adjacent the second angled wall **484** completely disengages from the interior wall of the intermediate chamber **446**, thereby exposing a second gap between the interior wall of the intermediate chamber **446** and the second angled wall **484**. As the variable flow head **474** continues to be retracted up to the second, fully-retracted position, the first gap and the second gap continue to increase in size, thereby increasing the agent flow from the inlet chamber **426** into the intermediate chamber **446** and the outlet chamber **448**.

As shown in FIGS. **8B** and **8C**, the integrated check valve **490** is positioned within the outlet chamber **448**. According to an exemplary embodiment, the integrated check valve **490** is configured to prevent agent, water, and/or an agent-water solution from flowing through the valve spout outlet **452** up the valve spout **440** into the intermediate chamber **446** and/or the inlet chamber **426**. As shown in FIGS. **8B** and **8C**, the integrated check valve **490** includes (i) a base, shown base **492**, that extends along the center of the valve spout **440** and entirely across the outlet chamber **448**, and (ii) a pair of pivotal blockers, shown as flaps **494**, extending in opposing directions from the base **492** at a downward angle to the interior wall of the outlet chamber **448**. The flaps **494** are pivotally coupled to the interior wall of the outlet chamber **448** with couplers, shown as pivotal couplers **496**. According to an exemplary embodiment, agent flow from the intermediate chamber **446** to the outlet chamber **448** forces the flaps **494** downward such that the flaps **494** pivot away from the base **492**, opening the integrated check valve **490**. Conversely, agent, water, and/or an agent-water solution flowing in the opposing direction forces the flaps **494** upward such that the flaps **494** pivot toward the base **492**, closing the integrated check valve **490**.

According to an exemplary embodiment, the agent check valve **132** is configured to prevent agent, water, and/or an agent-water solution from flowing back into the agent circuit **120**. Therefore, only agent may flow through the agent check valve **132** towards the ratio controller **140**, but nothing may flow through the agent check valve **132** in the reverse direction. In some embodiments, the agent circuit **120** does not include the agent check valve **132** (e.g., in embodiments that include the combined agent metering and shut-off valve assembly **400**, etc.).

As shown in FIGS. **3** and **4**, the ratio controller **140** is positioned to receive water from the water circuit **110** and/or agent from the agent circuit **120**. According to an exemplary embodiment, the ratio controller **140** is configured to facilitate mixing the water and the agent received thereby to provide an agent-water solution having a desired agent-to-water ratio.

As shown in FIGS. **5A-5D**, the ratio controller **140** includes a main body, shown as housing **142**. The housing **142** has a first side, shown as inlet side **144**, and an opposing second side, shown as outlet side **146**, spaced apart by a

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peripheral sidewall. As shown in FIGS. **5A**, **5C**, and **5D**, a protrusion, shown as diffuser **158**, extends from the outlet side **146** of the housing **142**. According to the exemplary embodiment shown in FIG. **5D**, the housing **142** and the diffuser **158** are integrally formed. As shown in FIG. **5D**, the housing **142** defines an internal cavity, shown a mixing chamber **148**. The inlet side **144** of the housing **142** defines an aperture, shown as water inlet **150**. According to an exemplary embodiment, the water inlet **150** is configured to couple to the water circuit **110** and receive water therefrom. As shown in FIG. **5D**, the ratio controller **140** includes a choke, shown as water nozzle **152**, coupled to an interior of the housing **142**, proximate the water inlet **150** and extending at least partially into the mixing chamber **148** (e.g., the water nozzle **152** is disposed entirely within the housing **142**, etc.). The water nozzle **152** has an inlet, shown as water inlet **154**, positioned to receive water from the water inlet **150** of the housing **142** and an outlet, shown as water outlet **156**.

As shown in FIGS. **5A-5D**, the ratio controller **140** includes agent inlets, shown as lower agent port **166** and upper agent port **168**. According to an exemplary embodiment, the lower agent port **166** and the upper agent port **168** are configured to couple to the agent circuit **120** and receive agent therefrom such that agent is injected into the mixing chamber **148** of the housing **142**. As shown in FIG. **5D**, the diffuser **158** has an inlet, shown as solution inlet **160**, and an outlet, shown as solution outlet **162**. The solution inlet **160** extends at least partially into the mixing chamber **148** of the housing **142**. The water outlet **156** of the water nozzle **152** and the solution inlet **160** of the diffuser **158** are thereby spaced a distance apart that forms a gap, shown as gap **164**, therebetween that has a width that is less than the width of the mixing chamber **148**. According to an exemplary embodiment, the agent flowing into the mixing chamber **148** through the lower agent port **166** and/or the upper agent port **168** mixes with the water exiting the water outlet **156** of the water nozzle **152**, and then discharges as an agent-water solution through the solution outlet **162** of the diffuser **158**.

As shown in FIG. **5D**, the peripheral sidewall of the housing **142** defines a first port, shown as high pressure port **170**, positioned proximate the water inlet **154** of the water nozzle **152** and a second port, shown as low pressure port **172**, positioned within the mixing chamber **148** (e.g., proximate the inlet side **144** of the housing **142**, etc.). As shown in FIGS. **5A**, **5B**, and the ratio controller **140** includes a manifold, shown as pressure manifold **174**, coupled to the housing **142**. As shown in FIGS. **5A** and **5D**, the pressure manifold **174** defines a first chamber, shown as high pressure chamber **176**, positioned to align with the high pressure port **170** and a second chamber, shown as low pressure chamber **178**, positioned to align with the low pressure port **172**. According to an exemplary embodiment, the high pressure port **170** and the high pressure chamber **176** facilitate monitoring the pressure of the water entering the ratio controller **140** (e.g., a high pressure, etc.) and the low pressure port **172** and the low pressure chamber **178** facilitate monitor the pressure of the solution within the mixing chamber **148** (e.g., a low pressure, etc.).

As shown in FIGS. **3** and **4**, the ratio controller **140** optionally includes a flow meter, shown as water flow meter **180**. The ratio controller **140** may therefore have an integrated water flow meter. According to an exemplary embodiment, the water nozzle **152** and the diffuser **158** function as a venturi (e.g., the water nozzle tapers inwards and the diffuser tapers outwards which causes the Venturi effect, a pressure drop as the velocity increases through the

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nozzle, etc.). According to an exemplary embodiment, the water flow meter **180** is coupled to the pressure manifold **174** such that the water flow meter **180** is configured to monitor the high pressure of the high pressure port **170** and the low pressure of the low pressure port **172**. The water flow meter **180** may be further configured to receive an indication of and/or determine the agent flow rate entering the mixing chamber **148**. In some embodiments, the indication of the agent flow rate may be provided by a signal from the agent circuit sensor **104** (e.g., an agent flow meter, etc.). In some embodiments, the water flow meter **180** is configured to determine the agent flow rate based on (i) the pressure of the agent exiting the agent pump **124** (e.g., received from the agent circuit sensor **104**, received directly from the agent pump **124**, etc.) and (ii) the current setting of the agent metering valve **126**, the combined agent metering and shut-off valve assembly **200**, or the combined agent metering and shut-off valve assembly **400** (e.g., the orifice size, valve position, etc.). According to an exemplary embodiment, the water flow meter **180** is configured to determine the flow rate of the water entering the ratio controller **140** based on the high pressure, the low pressure, and/or the agent flow rate (e.g., which may be used by the valve controller **128**, etc.).

According to an exemplary embodiment, the discharge valve **182** is configured to facilitate selectively restricting the flow of the agent-water solution. By way of example, the discharge valve **182** may (i) prevent the agent-water solution from passing therethrough when arranged in a first configuration (e.g., a closed configuration, etc.) and (ii) allow the agent-water solution to pass freely therethrough when arranged in a second configuration (e.g., an open configuration, etc.) such that the agent-water solution may be discharged from the fluid delivery system **100**. According to an exemplary embodiment, the agent-water solution exiting the discharge valve **182** is directed to one or more outlets of the fire fighting vehicle **10** such as a turret **190**, a structural discharge, and/or a hose reel. As shown in FIG. 1, the turret **190** is positioned on the front end of the front cabin **20**. As shown in FIG. 2, the turret **190** is positioned on the distal end of the aerial ladder assembly **50**.

As shown in FIGS. 1 and 2, the fire fighting vehicle **10** includes a control system, shown as pump engagement system **300**. As shown in FIG. 10, the pump engagement system includes a controller **310**. In one embodiment, the controller **310** is configured to selectively engage, selectively disengage, control, or otherwise communicate with components of the fire fighting vehicle **10**. As shown in FIG. 10, the controller **310** is coupled to a remote pump engage switch **320**, a user interface **330**, a pump engaged light **340**, a pump transfer case shift solenoid **350**, a transmission **360** (e.g., of the powertrain **40**, etc.), a pump transfer case **362** (e.g., of the powertrain **40**, etc.), and a parking brake **364**. The controller **310** may be configured to facilitate an operator in shifting the water pump **114** into a pump mode while in the front cabin **20** (e.g., using the user interface **330**, etc.) and/or remotely from any position on the fire fighting vehicle other than the front cabin **20** (e.g., using the remote pump engage switch **320**, etc.). By way of example, the controller **310** may send and receive signals with the remote pump engage switch **320**, the user interface **330**, the pump engaged light **340**, the pump transfer case shift solenoid **350**, the transmission **360**, the pump transfer case **362**, and/or the parking brake **364**.

The controller **310** may be implemented as a general-purpose processor, an application specific integrated circuit (ASIC), one or more field programmable gate arrays (FP-

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GAs), a digital-signal-processor (DSP), circuits containing one or more processing components, circuitry for supporting a microprocessor, a group of processing components, or other suitable electronic processing components. According to the exemplary embodiment shown in FIG. 10, the controller **310** includes a processing circuit **312** and a memory **314**. The processing circuit **312** may include an ASIC, one or more FPGAs, a DSP, circuits containing one or more processing components, circuitry for supporting a microprocessor, a group of processing components, or other suitable electronic processing components. In some embodiments, the processing circuit **312** is configured to execute computer code stored in the memory **314** to facilitate the activities described herein. The memory **314** may be any volatile or non-volatile computer-readable storage medium capable of storing data or computer code relating to the activities described herein. According to an exemplary embodiment, the memory **314** includes computer code modules (e.g., executable code, object code, source code, script code, machine code, etc.) configured for execution by the processing circuit **312**. In some embodiments, the controller **310** may represent a collection of processing devices (e.g., servers, data centers, etc.). In such cases, the processing circuit **312** represents the collective processors of the devices, and the memory **314** represents the collective storage devices of the devices.

According to an exemplary embodiment, the remote pump engage switch **320** is positioned remotely from the front cabin **20** of the fire fighting vehicle **10**. The remote pump engage switch **320** may be positioned on or at any location of the fire fighting vehicle **10** other than the front cabin **20**. Typically, if there is no need for fire extinguishing capabilities at a scene, a fire fighter will not activate a pump system of a fire fighting vehicle. In traditional systems, if a need for fire suppression arises after arrival, a mid-ship pump can only be shifted into a pump mode from inside the cab of the vehicle, which causes unnecessary delays. The remote pump engage switch **320** is positioned externally from the front cabin **20** such that the mid-ship pump (e.g., the water pump **114**, etc.) may be engaged without having to enter the front cabin **20**, saving valuable time and effort.

In one embodiment, the user interface **330** includes a display and an operator input. The display and/or the operator input may be positioned within the front cabin **20** and/or at any position along the exterior of the fire fighting vehicle **10**. The display may be configured to display a graphical user interface, an image, an icon, or still other information. In one embodiment, the display includes a graphical user interface configured to provide general information about the vehicle (e.g., vehicle speed, fuel level, warning lights, agent levels, water levels, etc.). The graphical user interface may also be configured to display a current water flow rate, a current agent flow rate, a current agent-to-water ratio, etc. By way of example, the graphical user interface may be configured to provide specific information regarding the operation of the fire fighting vehicle **10**, the fluid delivery system **100**, and/or the pump engagement system **300**.

The operator input may be used by an operator to provide commands to at least one of the fire fighting vehicle **10**, the fluid delivery system **100** (e.g., the water pump **114**, the agent pump **124**, the valve controller **128**, the agent shut-off valve **130**, the water flow meter **180**, the discharge valve, etc.), and the pump engagement system **300** (e.g., the pump engaged light **340**, the transmission **360**, the pump transfer case **362**, the parking brake **364**, the pump transfer case shift solenoid **350**, etc.). The operator input may include one or

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more buttons, knobs, touchscreens, switches, levers, joysticks, pedals, or handles. The operator may be able to manually control some or all aspects of the operation of the pump engagement system 300, the fluid delivery system 100, and/or the fire fighting vehicle 10 using the display and the operator input. It should be understood that any type of display or input controls may be implemented with the systems and methods described herein.

According to an exemplary embodiment, the controller 310 is configured to receive a pump shift input. In some embodiments, the pump shift input is provided by a user with the remote pump engage switch 320 (e.g., externally from the front cabin 20, etc.). In some embodiments, the pump shift input is provided by a user with the user interface 330 (e.g., externally from the front cabin 20, internally within the front cabin 20, etc.). The controller 310 is further configured to receive (i) a transmission gear signal from the transmission 360 such that the controller 310 may determine whether the transmission 360 is in neutral and (ii) a parking brake signal from the parking brake 364 such that the controller 310 may determine whether the parking brake 364 is engaged in response to receiving the pump shift input. In some embodiments, the controller 310 is configured to shift the transmission 360 into neutral in response to the transmission 360 being in gear (e.g., reverse, drive, etc.). In some embodiments, the controller 310 is configured to provide an indication on the user interface 330 that the transmission 360 needs to be shifted into neutral by the operator in response to the transmission 360 being in gear. In some embodiments, the controller 310 is configured to engage the parking brake 364 in response to the parking brake 364 not being engaged. In some embodiments, the controller 310 is configured to provide an indication on the user interface 330 that the parking brake 364 needs to be engaged by an operator in response to the parking brake 364 not being engaged.

According to an exemplary embodiment, the controller 310 is configured to send a shift signal to the pump transfer case shift solenoid 350 such that the pump transfer case 362 may be shifted into the pump mode in response to the transmission 360 being in neutral and the parking brake being engaged. According to an exemplary embodiment, the pump transfer case 362 is configured to selectively, mechanically couple the engine of the powertrain 40 to the water pump 114 such that the water pump 114 may be selectively driven by the engine (e.g., during the pump mode, etc.). By way of example, the pump transfer case shift solenoid 350 engages a shift element, shown as shift cylinder 352, in response to receiving the shift signal from the controller 310. The engagement of the shift cylinder 352 with the pump transfer case shift solenoid 350 causes the shift cylinder 352 to shift the pump transfer case 362 from a first mode (e.g., a non-pumping mode, etc.) where the engine is effectively decoupled from the water pump 114 to a second mode (e.g., the pump mode, etc.) where the engine is effectively coupled to the water pump 114. When in the second, pump mode, the engine may thereby drive the water pump 114 through the pump transfer case 362.

The controller 310 may be further configured to determine whether the pump transfer case 362 was effectively shifted into the second, pump mode after the engagement of the shift cylinder 352. The controller 310 may be configured to provide an indication on the user interface 330 that the shift failed in response to the pump transfer case 362 not being in the pump mode. The controller 310 may be configured to shift the transmission 360 into drive such that the engine begins to drive the water pump 114 in response to the pump transfer case 362 shifting into the pump mode. In some

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embodiments, the controller 310 is configured to provide an indication that the water pump 114 has been engaged and is in operation at least one of on the user interface 330 and with the pump engaged light 340 (e.g., illuminating the pump engaged light 340, etc.). Thereafter, the operator may discharge water, agent, and/or an agent-water solution using the fluid delivery system 100 to suppress and extinguish a fire.

Referring now to FIG. 11, a method 1100 for a shifting a pump into a pump mode is shown according to an exemplary embodiment. At step 1102, a controller (e.g., the controller 310, etc.) is configured to receive a pump shift input. In some embodiments, the pump shift input is provided by a user with a pump switch (e.g., the remote pump engage switch 320, etc.). In some embodiments, the pump shift input is provided by a user with a user interface (e.g., the user interface 330, etc.). At step 1104, the controller is configured to determine whether a transmission (e.g., the transmission 360, etc.) of a vehicle (e.g., the fire fighting vehicle 10, etc.) is in neutral. At step 1106, the controller is configured to shift the transmission into neutral or provide an indication (e.g., on the user interface 330, etc.) that the transmission needs to be shifted into neutral to proceed in response to the transmission being in gear (e.g., not in neutral, etc.). At step 1108, the controller is configured to determine whether a parking brake (e.g., the parking brake 364, etc.) is engaged in response to the transmission being in neutral. At step 1110, the controller is configured to engage the parking brake or provide an indication (e.g., on the user interface 330, etc.) that the parking brake needs to be engaged to proceed in response to the parking brake not being engaged. At step 1112, the controller is configured to shift a pump transfer case (e.g., the pump transfer case shift solenoid 350 coupled to the pump transfer case 362, etc.) coupled to a pump (e.g., the water pump 114, etc.) and an engine of the vehicle into a pump mode such that the pump may be driven by the engine in response to the transmission being in neutral and the parking brake being engaged.

At step 1114, the controller is configured to determine whether the pump transfer case shifted into the pump mode. At step 1116, the controller is configured to provide an indication (e.g., on the user interface 330, etc.) that the shift failed in response to the pump transfer case not being in the pump mode. At step 1118, the controller is configured to shift the transmission into drive such that the engine begins to drive the pump in response to the transfer case shifting into the pump mode. At step 1120, the controller is configured to provide an indication that the pump is engaged (e.g., on the user interface 330, with the pump engaged light 340, etc.).

As utilized herein, the terms “approximately”, “about”, “substantially”, and similar terms are intended to have a broad meaning in harmony with the common and accepted usage by those of ordinary skill in the art to which the subject matter of this disclosure pertains. It should be understood by those of skill in the art who review this disclosure that these terms are intended to allow a description of certain features described and claimed without restricting the scope of these features to the precise numerical ranges provided. Accordingly, these terms should be interpreted as indicating that insubstantial or inconsequential modifications or alterations of the subject matter described and claimed are considered to be within the scope of the invention as recited in the appended claims.

It should be noted that the term “exemplary” as used herein to describe various embodiments is intended to indicate that such embodiments are possible examples, representations, and/or illustrations of possible embodi-

ments (and such term is not intended to connote that such embodiments are necessarily extraordinary or superlative examples).

The terms “coupled,” “connected,” and the like, as used herein, mean the joining of two members directly or indirectly to one another. Such joining may be stationary (e.g., permanent, etc.) or moveable (e.g., removable, releasable, etc.). Such joining may be achieved with the two members or the two members and any additional intermediate members being integrally formed as a single unitary body with one another or with the two members or the two members and any additional intermediate members being attached to one another.

References herein to the positions of elements (e.g., “top,” “bottom,” “above,” “below,” “between,” etc.) are merely used to describe the orientation of various elements in the figures. It should be noted that the orientation of various elements may differ according to other exemplary embodiments, and that such variations are intended to be encompassed by the present disclosure.

Also, the term “or” is used in its inclusive sense (and not in its exclusive sense) so that when used, for example, to connect a list of elements, the term “or” means one, some, or all of the elements in the list. Conjunctive language such as the phrase “at least one of X, Y, and Z,” unless specifically stated otherwise, is otherwise understood with the context as used in general to convey that an item, term, etc. may be either X, Y, Z, X and Y, X and Z, Y and Z, or X, Y, and Z (i.e., any combination of X, Y, and Z). Thus, such conjunctive language is not generally intended to imply that certain embodiments require at least one of X, at least one of Y, and at least one of Z to each be present, unless otherwise indicated.

It is important to note that the construction and arrangement of the lateral access limitation system as shown in the exemplary embodiments is illustrative only. Although only a few embodiments of the present disclosure have been described in detail, those skilled in the art who review this disclosure will readily appreciate that many modifications are possible (e.g., variations in sizes, dimensions, structures, shapes and proportions of the various elements, values of parameters, mounting arrangements, use of materials, colors, orientations, etc.) without materially departing from the novel teachings and advantages of the subject matter recited. For example, elements shown as integrally formed may be constructed of multiple parts or elements. It should be noted that the elements and/or assemblies of the components described herein may be constructed from any of a wide variety of materials that provide sufficient strength or durability, in any of a wide variety of colors, textures, and combinations. Accordingly, all such modifications are intended to be included within the scope of the present inventions. Other substitutions, modifications, changes, and omissions may be made in the design, operating conditions, and arrangement of the preferred and other exemplary embodiments without departing from scope of the present disclosure or from the spirit of the appended claims.

The invention claimed is:

1. A ratio controller for a fluid system of a fire apparatus, the ratio controller comprising:

a housing having a unitary structure, the housing defining:
a water inlet configured to receive water from a water supply;

an outlet configured to output an agent-water solution;
a mixing chamber positioned between the water inlet and the outlet;

a first pressure port extending through a sidewall of the housing at a first position proximate the water inlet; and

a second pressure port extending through the sidewall of the housing at a second position proximate the mixing chamber; and

an agent port positioned between the water inlet and the outlet, the agent port protruding radially outward from an exterior surface of the sidewall external to the ratio controller, the agent port defining an agent inlet extending through the housing to the mixing chamber, the agent port configured to receive agent from an agent supply and provide the agent to the mixing chamber.

2. The ratio controller of claim 1, further comprising a nozzle extending at least partially into the mixing chamber, the nozzle including a nozzle inlet positioned proximate the first pressure port and a nozzle outlet positioned within the mixing chamber.

3. The ratio controller of claim 2, wherein the first pressure port is positioned between the water inlet and the nozzle inlet.

4. The ratio controller of claim 1, further comprising a diffuser extending from the outlet outward from the housing, the diffuser including a diffuser inlet positioned within the mixing chamber and a diffuser outlet.

5. The ratio controller of claim 1, further comprising a nozzle and a diffuser, wherein at least one of the nozzle or the diffuser at least partially extends into the mixing chamber such that a nozzle outlet of the nozzle and a diffuser inlet of the diffuser are spaced a distance that is less than a width of the mixing chamber.

6. The ratio controller of claim 1, wherein the agent port is a first agent port and the agent inlet is a first agent inlet, further comprising a second agent port positioned between the water inlet and the outlet, the second agent port protruding radially outward from the exterior surface of the housing external to the ratio controller, wherein the second agent port defines a second agent inlet extending through the housing to the mixing chamber and configured to receive the agent from the agent supply.

7. The ratio controller of claim 6, wherein the first agent port and the second agent port are positioned along the sidewall of the housing.

8. The ratio controller of claim 7, wherein the first agent port and the second agent port are positioned on opposing sides of the mixing chamber, and wherein the first agent port and the second agent port are integrally formed with the housing.

9. The ratio controller of claim 1, further comprising a pressure manifold coupled to an exterior of the housing, wherein the pressure manifold defines a first chamber positioned to align with the first pressure port and a second chamber positioned to align with the second pressure port.

10. The ratio controller of claim 1, further comprising a flow meter coupled to the first pressure port and the second pressure port to monitor (i) an inlet pressure of the water entering the ratio controller and (ii) an intermediate pressure within the mixing chamber to facilitate determining a water flow rate of the water.