US011713933B2

(12) **United States Patent**
Kolev et al.

(10) **Patent No.: US 11,713,933 B2**
(45) **Date of Patent: *Aug. 1, 2023**

(54) **SEMIAUTOMATIC FIREARM**

(71) Applicant: **Savage Arms, Inc.**, Westfield, MA (US)

(72) Inventors: **Ivan Kolev**, Suffield, CT (US); **John Linscott**, Easthampton, MA (US)

(73) Assignee: **Savage Arms, Inc.**, Westfield, MA (US)

(*) Notice: Subject to any disclaimer, the term of this patent is extended or adjusted under 35 U.S.C. 154(b) by 0 days.

This patent is subject to a terminal disclaimer.

(21) Appl. No.: **17/036,610**

(22) Filed: **Sep. 29, 2020**

(65) **Prior Publication Data**

US 2021/0148662 A1 May 20, 2021

Related U.S. Application Data

(63) Continuation of application No. 15/352,330, filed on Nov. 15, 2016, now Pat. No. 10,788,277, which is a (Continued)

(51) **Int. Cl.**
F41A 3/46 (2006.01)
F41A 3/66 (2006.01)
(Continued)

(52) **U.S. Cl.**
CPC ***F41A 3/46*** (2013.01); ***F41A 3/12*** (2013.01); ***F41A 3/66*** (2013.01); ***F41A 3/70*** (2013.01); ***F41A 3/72*** (2013.01); ***F41A 19/27*** (2013.01)

(58) **Field of Classification Search**
CPC F41A 3/14; F41A 3/44; F41A 3/46
See application file for complete search history.

(56) **References Cited**

U.S. PATENT DOCUMENTS

533,101 A *	1/1895	Tesch	....	F41A 17/66 89/18
1,043,670 A *	11/1912	Chronis	....	F41A 3/38 89/187.01

(Continued)

FOREIGN PATENT DOCUMENTS

CH	626717 A5	11/1981
DE	10200703 4672	1/2009

(Continued)

OTHER PUBLICATIONS

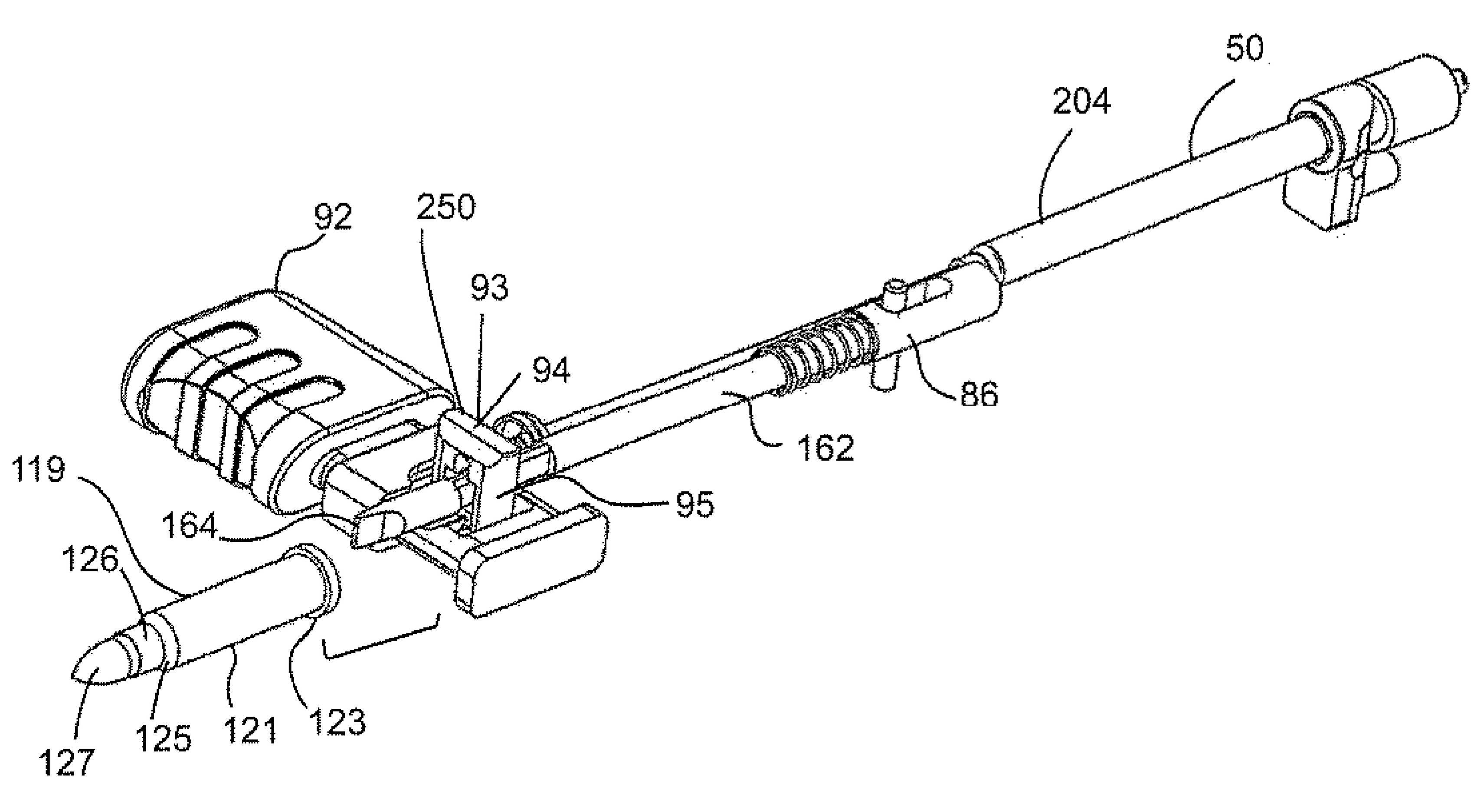
Guns & Ammo Network, "Innovative, Remarkable, Reliable . . . Getting Inside Savage's Accutrigger". Sep. 23, 2010. Available online at: http://www.shootingtimes.com/long-guns/longgun_reviews_savage_0813/ (7 pgs.).

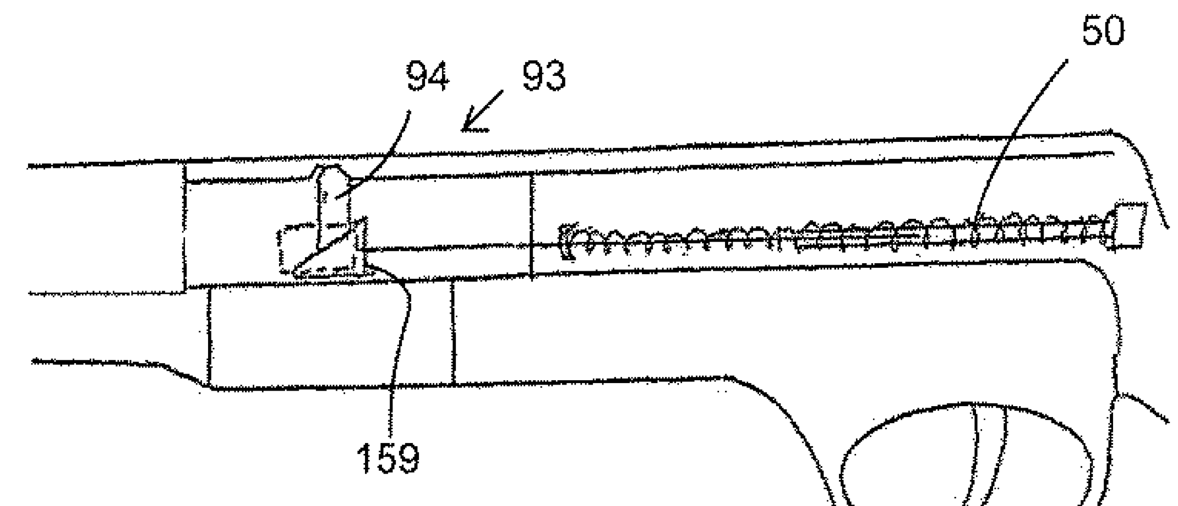
Primary Examiner — Derrick R Morgan
(74) *Attorney, Agent, or Firm* — Christensen, Fonder, Dardi & Herbert PLLC; Douglas J. Christensen

(57) **ABSTRACT**

A reciprocating bolt assembly has delayed blowback and a firing pin block. Features prevent out-of-battery firing, when the bolt assembly is not fully engaged to the firing chamber or barrel face, a movable member within a bolt body functions as a blocking member that blocks the firing pin and prevents the firing pin from striking a cartridge. In embodiments, the firing pin has two stop portions that the movable member can engage depending on the cycle status of the firearm. A reverse cam mechanism associated with the firing pin blocking provides a resistance to and delays blowback.

19 Claims, 62 Drawing Sheets





Related U.S. Application Data

continuation of application No. PCT/US2015/031210, filed on May 15, 2015, which is a continuation of application No. 14/599,199, filed on Jan. 16, 2015, now Pat. No. 9,599,417, and a continuation of application No. 14/599,408, filed on Jan. 16, 2015, now Pat. No. 9,513,076, and a continuation of application No. 14/599,396, filed on Jan. 16, 2015, now Pat. No. 9,810,496.

(60) Provisional application No. 61/993,569, filed on May 15, 2014, provisional application No. 61/993,563, filed on May 15, 2014, provisional application No. 61/993,541, filed on May 15, 2014.

(51) **Int. Cl.**
F41A 19/27 (2006.01)
F41A 3/72 (2006.01)
F41A 3/70 (2006.01)
F41A 3/12 (2006.01)

(56) **References Cited**

U.S. PATENT DOCUMENTS

1,410,270	A		3/1922	Pedersen	
1,656,961	A	*	1/1928	Soncini	F41A 17/66 89/149
1,737,974	A		12/1929	Pedersen	
2,089,671	A	*	8/1937	Stecke	F41A 3/46 89/187.01
2,270,683	A	*	1/1942	Janecek	F41A 3/56 42/20
2,453,830	A	*	11/1948	Chadwick	F41A 25/18 89/44.01
2,465,749	A	*	3/1949	Rataiczak	F41A 17/66 89/136
2,473,373	A		6/1949	Howell	
2,527,895	A	*	10/1950	Tassan	C01B 32/384 89/187.01
2,576,973	A	*	12/1951	Simpson	F41A 19/30 89/187.01
2,585,195	A		2/1952	Walker	
2,594,354	A	*	4/1952	Schnepel	F41A 19/30 89/193
2,601,808	A		7/1952	Clarke	
2,603,019	A		7/1952	Elkas	
2,626,474	A	*	1/1953	Lochhead	F41A 19/43 89/187.01
2,638,694	A		5/1953	Morris	
2,848,832	A	*	8/1958	Lee	F41A 17/72 42/70.08
2,873,546	A		2/1959	Allyn	
2,900,877	A	*	8/1959	McClenahan	F41A 3/82 89/173
2,912,779	A		11/1959	Catlin et al.	
2,921,502	A	*	1/1960	Amsler	F41A 19/43 89/149
2,962,936	A	*	12/1960	Muhlemann	F41A 5/18 89/190
2,975,680	A		3/1961	Wilson	
3,153,982	A	*	10/1964	Seifried	F41A 3/46 89/194
3,235,993	A		2/1966	Magardo	
3,292,492	A		12/1966	Sturtevant	
3,566,745	A	*	3/1971	Jauch	F41A 3/46 42/16
3,631,622	A		1/1972	Walker et al.	
3,738,223	A		6/1973	Post et al.	
3,848,510	A	*	11/1974	Wolpert	F41A 3/46 89/173
3,857,325	A		12/1974	Thomas	
3,893,369	A		7/1975	Benelli	
4,069,607	A		1/1978	Jurek	
4,203,243	A		5/1980	Hickman	
4,270,295	A	*	6/1981	Grehl	F41A 17/64 42/70.08
4,344,246	A	*	8/1982	Bauman	F41A 17/66 89/190
4,547,988	A		10/1985	Nilsson	
4,677,898	A	*	7/1987	Malhotra	F41A 3/46 89/187.01
4,815,356	A	*	3/1989	Hupp	F41A 3/40 384/282
4,922,640	A	*	5/1990	Toombs	F41B 11/00 42/16
5,157,209	A	*	10/1992	Dunn	F41A 19/32 89/154
5,259,137	A	*	11/1993	Blenk	F41A 3/42 42/16
5,267,407	A	*	12/1993	Bornancini	F41A 35/06 42/70.08
5,614,691	A	*	3/1997	Taylor	F41A 11/02 89/181
5,655,326	A		8/1997	Levavi et al.	
5,666,754	A	*	9/1997	de Oliveira Masina	F41A 17/72 42/70.08
5,726,376	A	*	3/1998	Menges	F41A 3/44 89/190
5,760,328	A		6/1998	Robbins	
5,900,576	A	*	5/1999	Gabriel	F41A 3/46 89/187.01
5,974,942	A		11/1999	Dionne	
6,256,918	B1	*	7/2001	Szabo	F41A 17/72 89/154
6,349,495	B1	*	2/2002	Fluhr	F41A 17/66 42/70.08
6,418,655	B1	*	7/2002	Kay	F41F 1/08 89/185
6,553,706	B1		4/2003	Gancarz et al.	
6,782,791	B2	*	8/2004	Moore	F41A 17/46 89/44.01
7,055,422	B1	*	6/2006	Dindl	F41A 3/54 42/1.06
7,204,051	B2	*	4/2007	Thomele	F41A 17/72 42/70.08
7,874,240	B2	*	1/2011	Akhavan	F41A 5/24 89/179
8,074,391	B2	*	12/2011	Petrůj	F41A 25/12 42/1.06
8,122,633	B2		2/2012	Kerbrat et al.	
8,220,193	B1		7/2012	Lynch	
8,250,799	B2		8/2012	Duperry et al.	
8,528,458	B2		9/2013	Windauer	
8,807,010	B2	*	8/2014	Bubits	F41C 3/00 89/187.01
9,488,431	B2	*	11/2016	Bubits	F41C 3/00
9,513,076	B2	*	12/2016	Kolev	F41A 3/46
9,664,466	B2	*	5/2017	Gryparis	F41F 1/00
9,964,369	B2	*	5/2018	Garrow	F41A 3/44
10,175,019	B1	*	1/2019	Al-Mutawa	F41A 17/72
10,895,424	B2	*	1/2021	Tertin	F41A 3/42
2003/0089014	A1	*	5/2003	Schuerman	F41A 3/42 42/16
2003/0150322	A1	*	8/2003	Barrett, Jr.	F41A 3/46 89/182
2005/0132875	A1	*	6/2005	Murello	F41A 3/46 89/198
2005/0188578	A1	*	9/2005	Engel	F41A 3/18 42/16
2005/0217473	A1	*	10/2005	Murello	F41A 3/40 89/187.01
2005/0235817	A1	*	10/2005	Murello	F41A 3/46 89/33.01
2005/0257682	A1	*	11/2005	Hajjar	F41A 3/46 89/188
2009/0308240	A1	*	12/2009	Kneppler	F41A 3/42 89/180

(56) **References Cited**

U.S. PATENT DOCUMENTS

2010/0024273	A1	2/2010	Duperry et al.	
2010/0186581	A1*	7/2010	Hajjar	F41A 3/46 89/188
2010/0313459	A1*	12/2010	Gomez	F41A 17/72 42/70.08
2011/0079137	A1	4/2011	Audibert et al.	
2013/0074390	A1	3/2013	Wössner et al.	
2013/0199069	A1	8/2013	Kerbrat et al.	
2014/0317983	A1*	10/2014	Bush	F42B 5/32 42/17
2015/0316335	A1*	11/2015	Withey	F41A 19/11 42/16
2015/0330727	A1*	11/2015	Kolev	F41A 3/70 42/16
2015/0330731	A1*	11/2015	Kolev	F41A 15/14 42/25
2016/0252316	A1*	9/2016	Garrow	F41A 19/30 89/181

FOREIGN PATENT DOCUMENTS

EP	0 017 506	10/1980
WO	WO 94/09334	4/1994

* cited by examiner

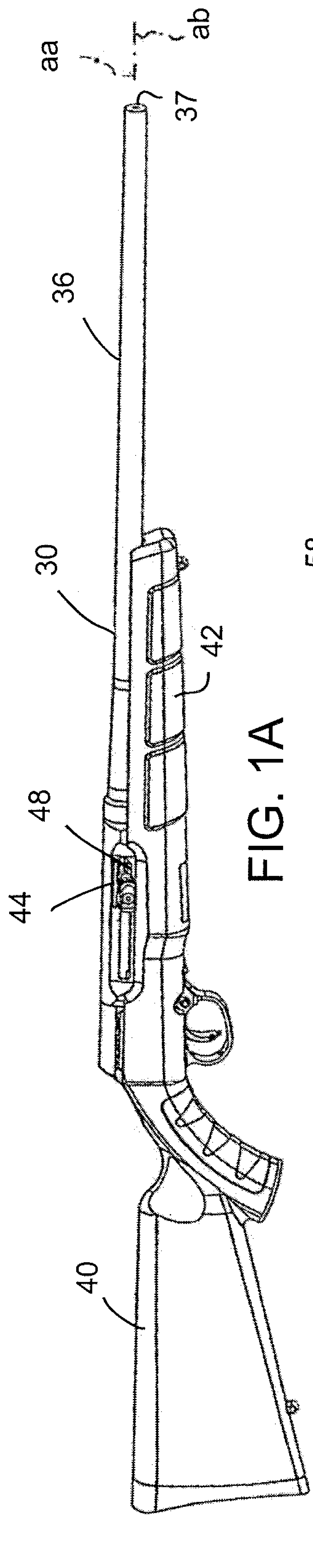
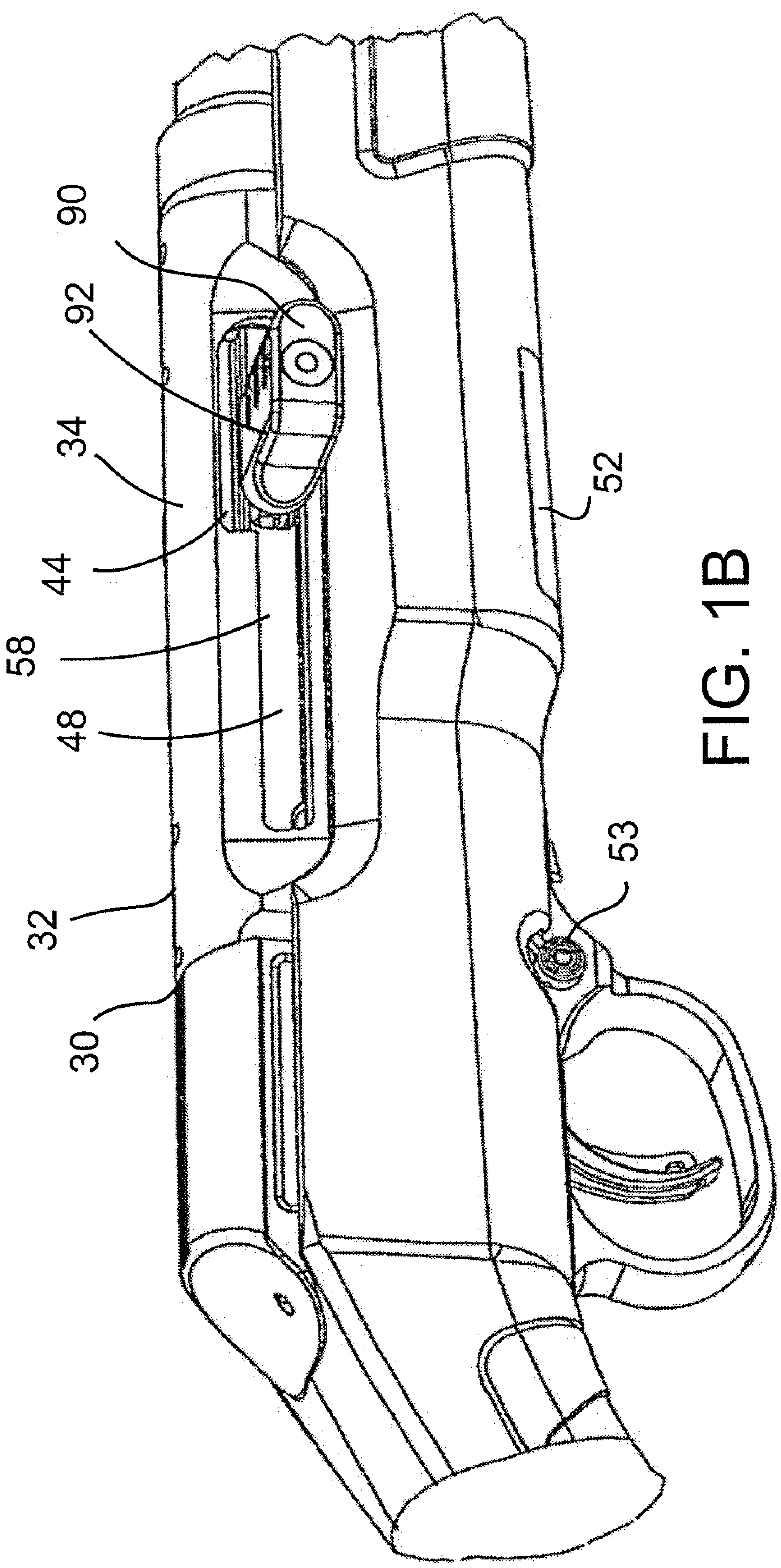


FIG. 1A



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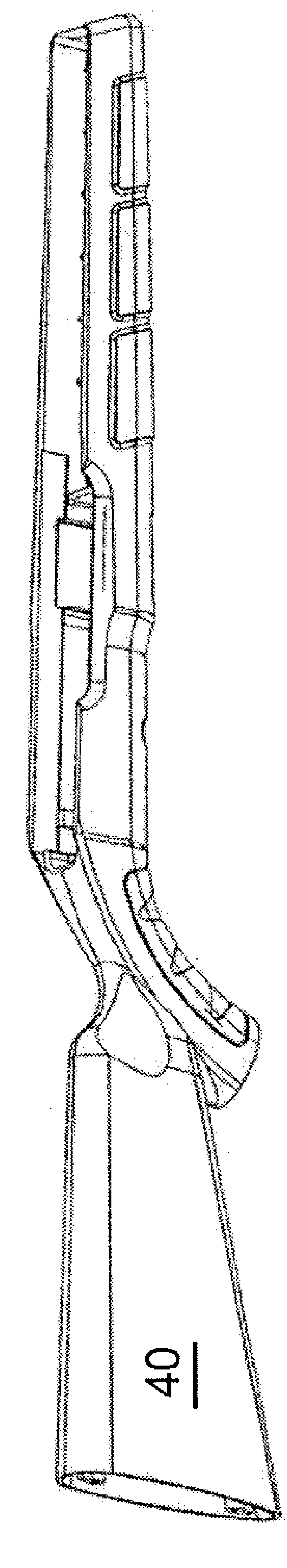

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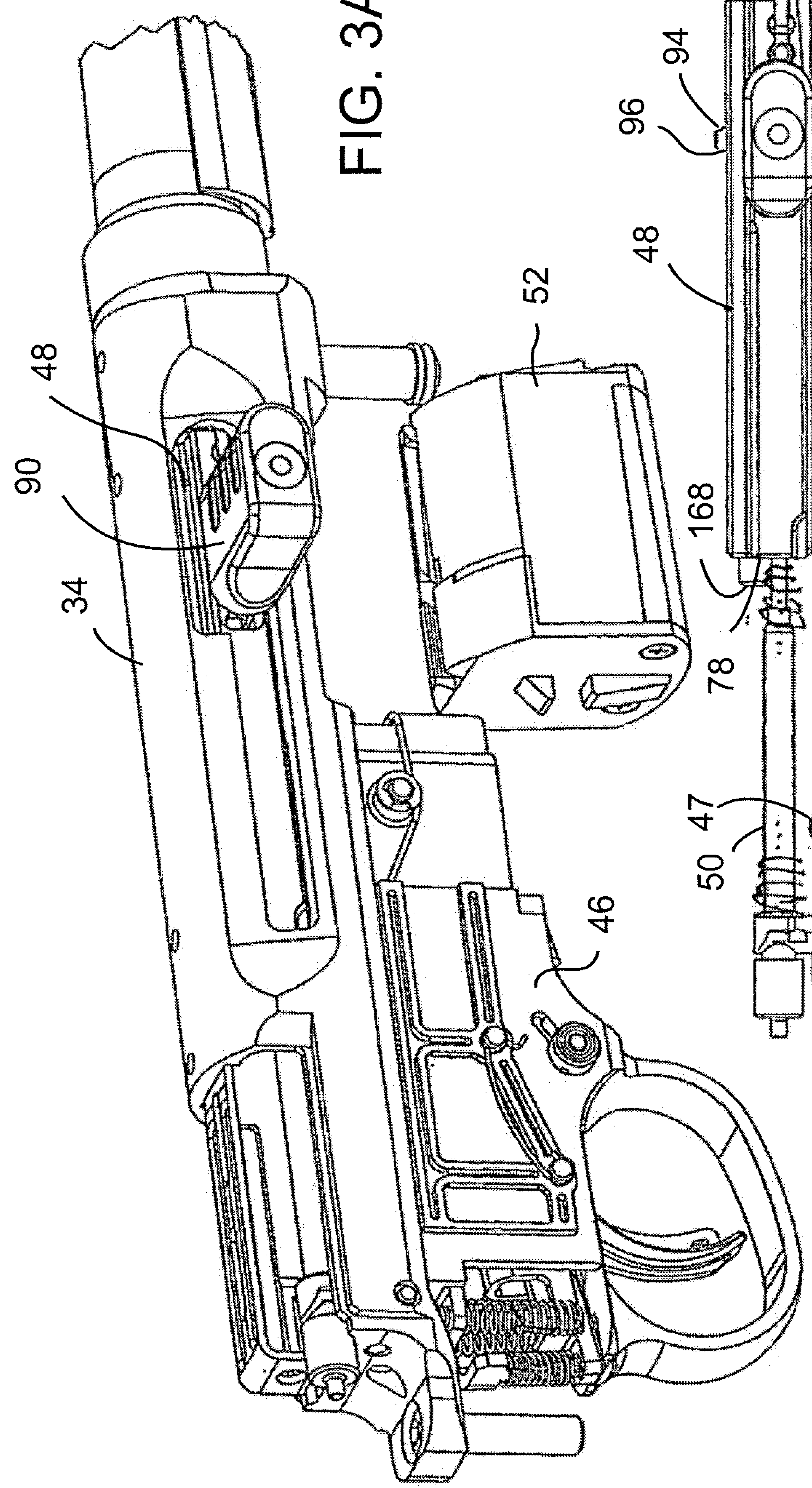

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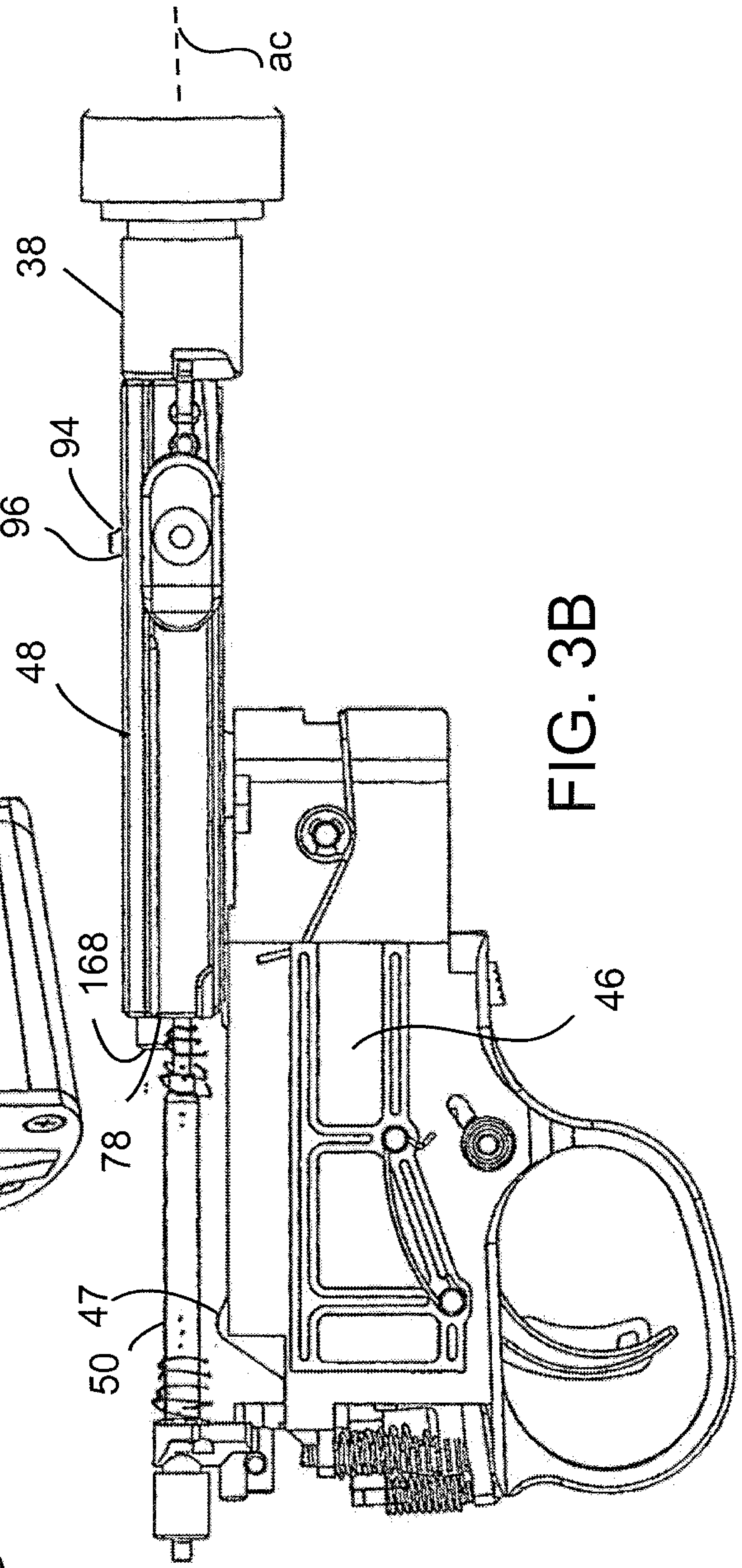

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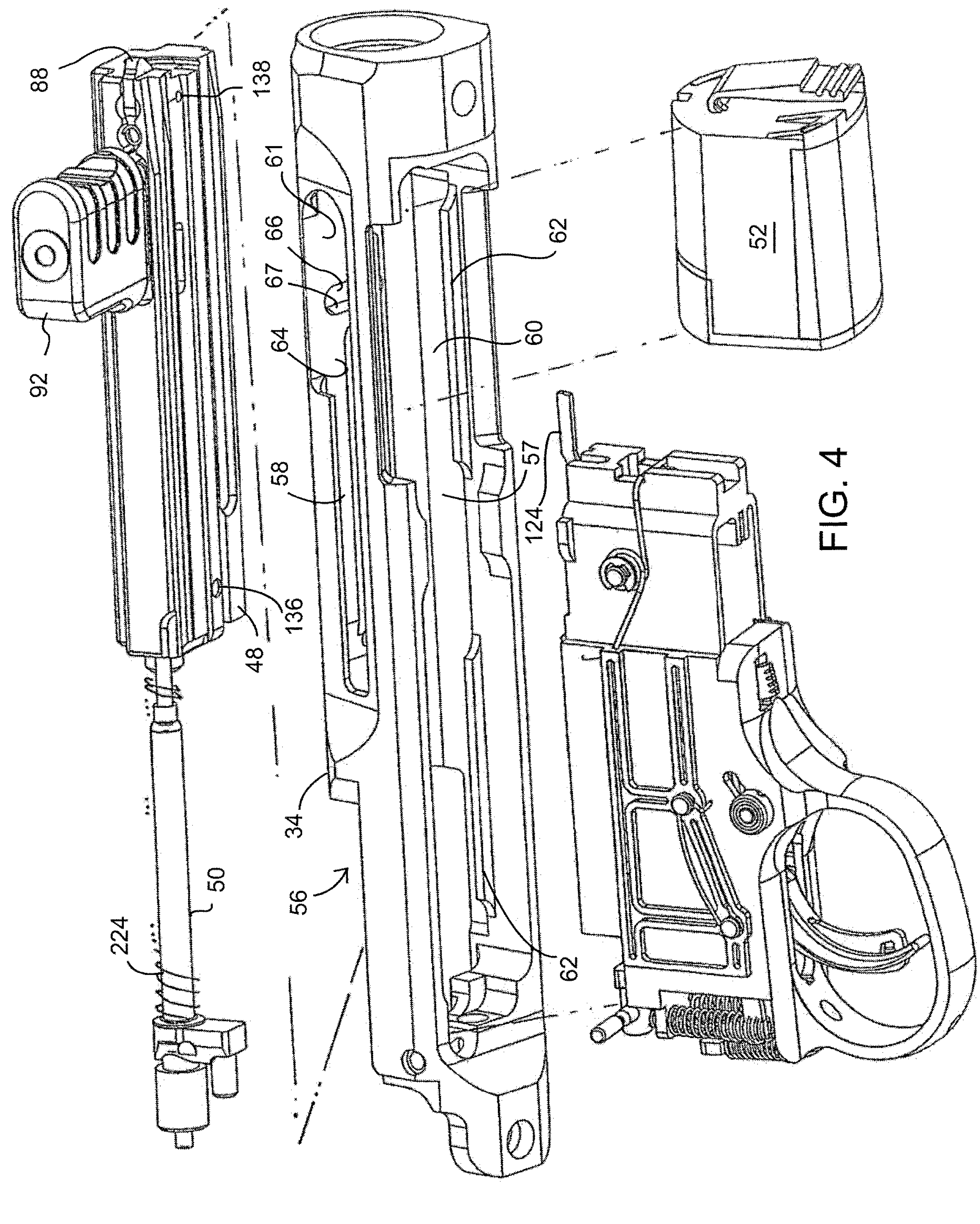

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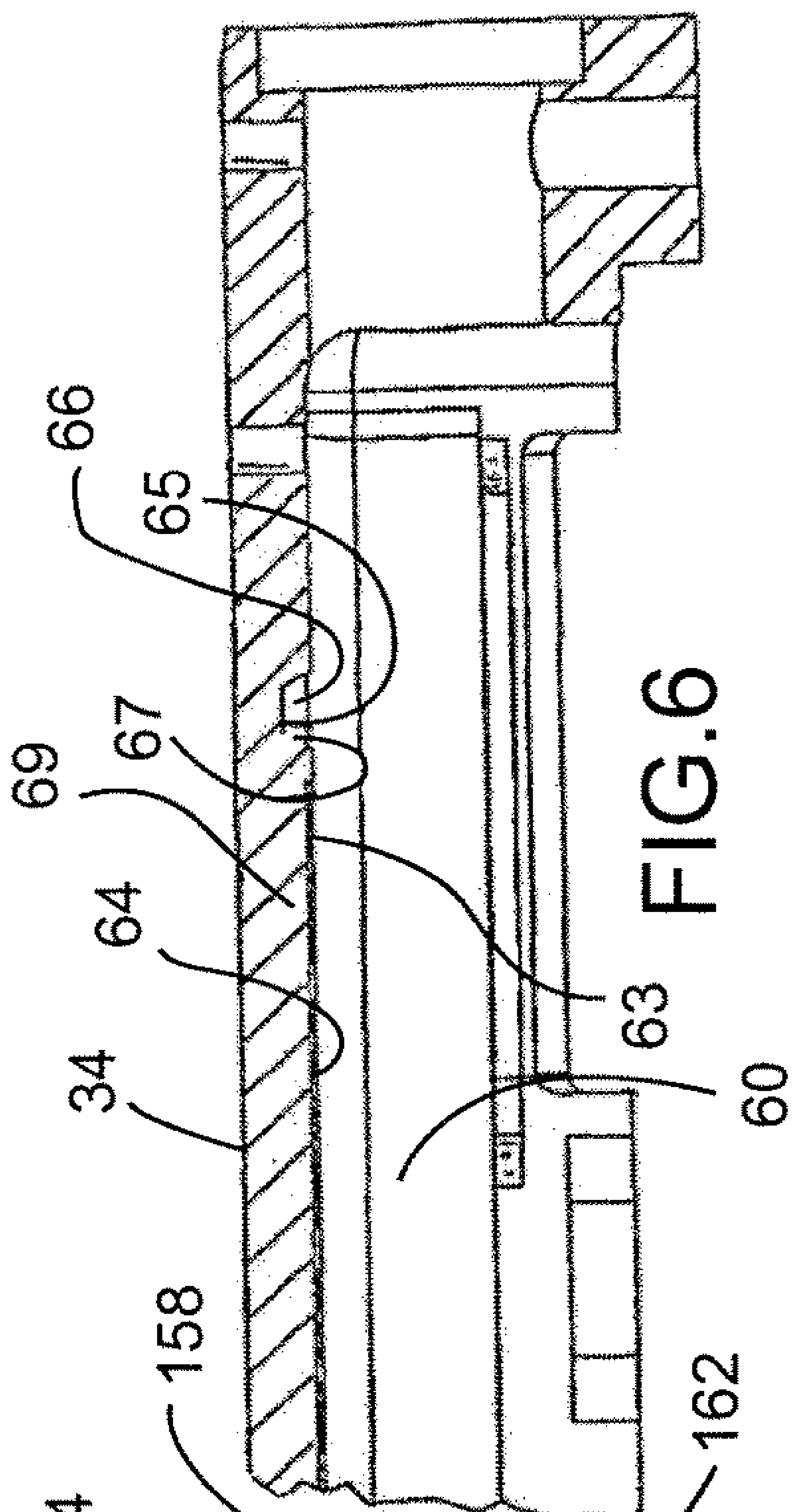
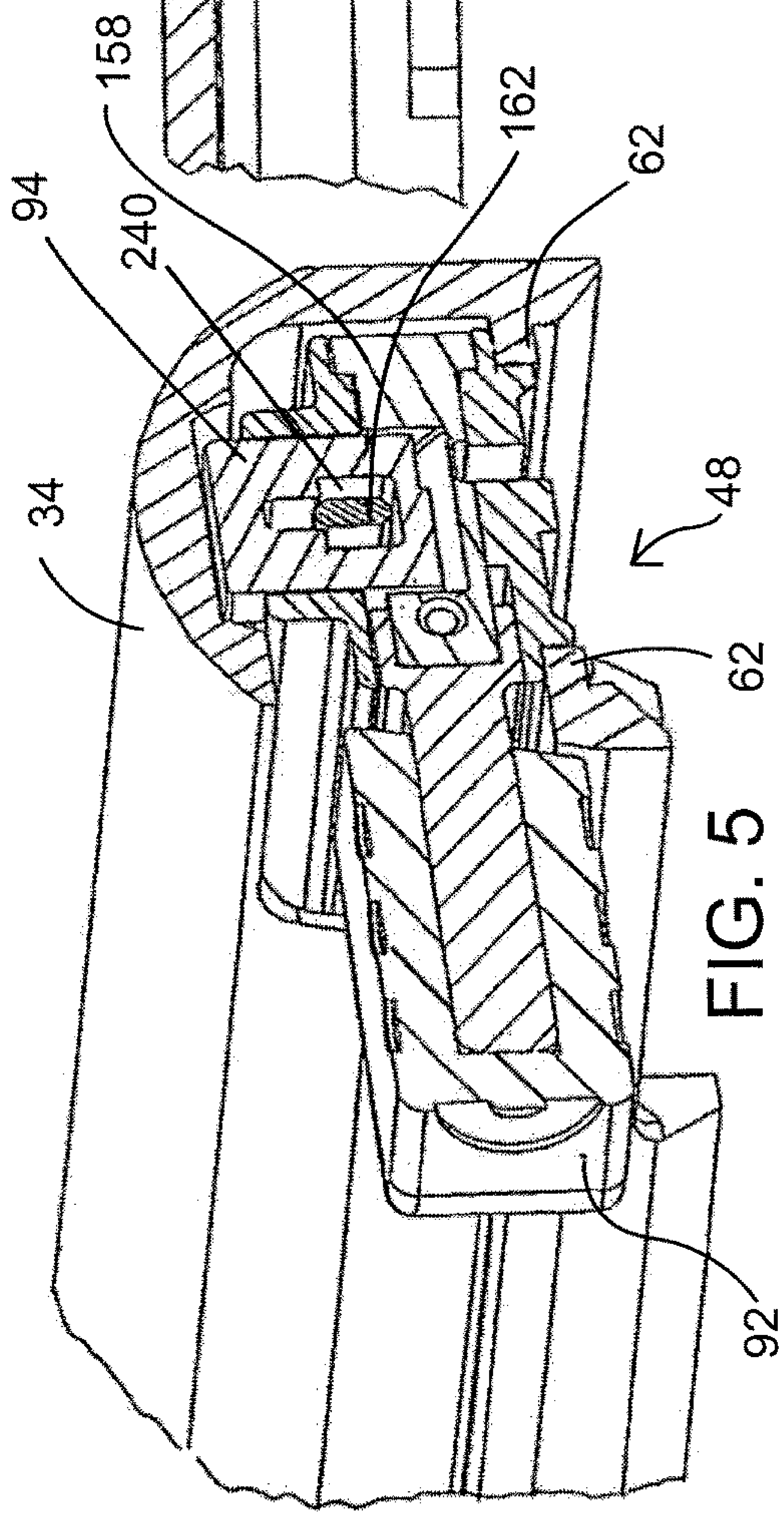


FIG.6



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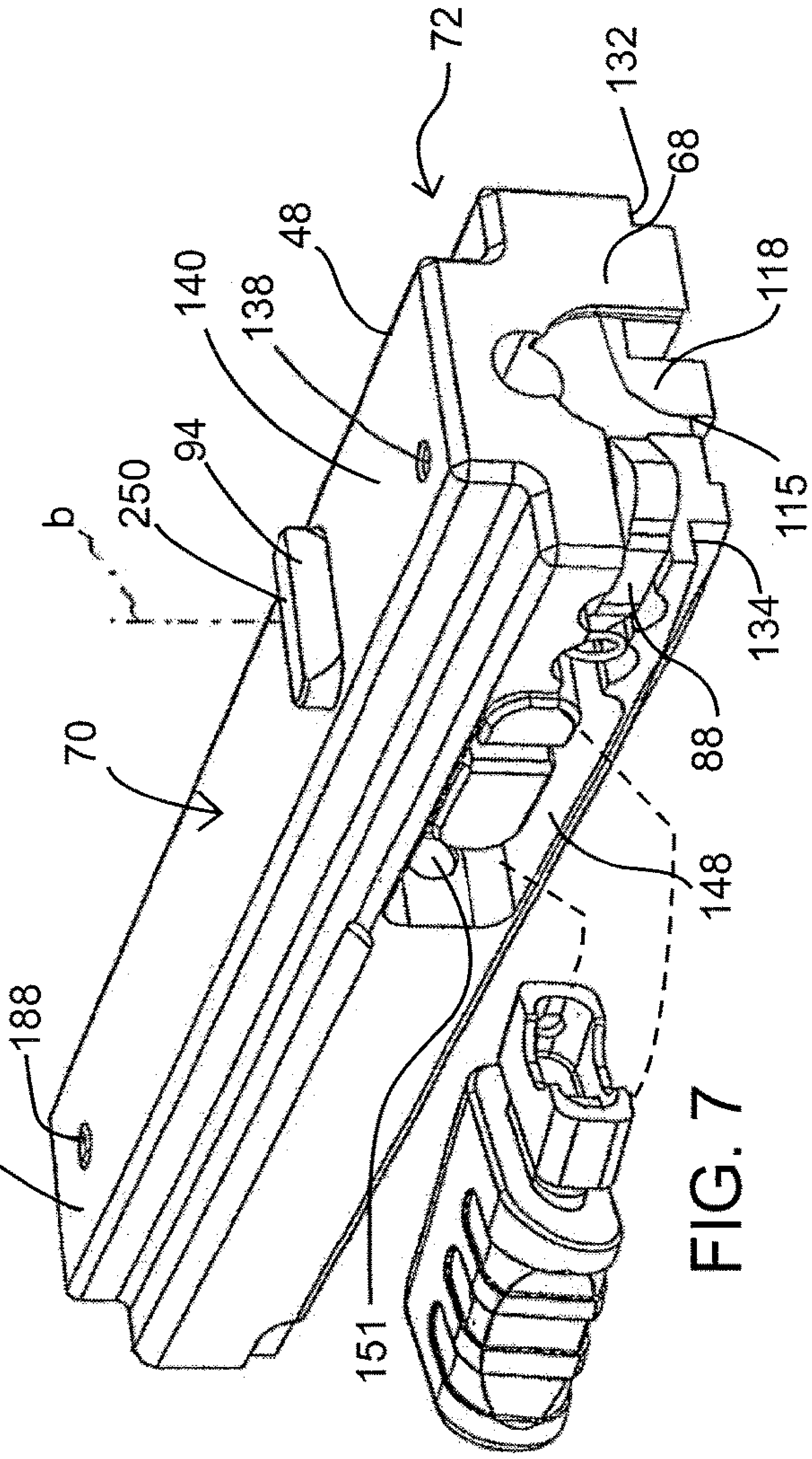

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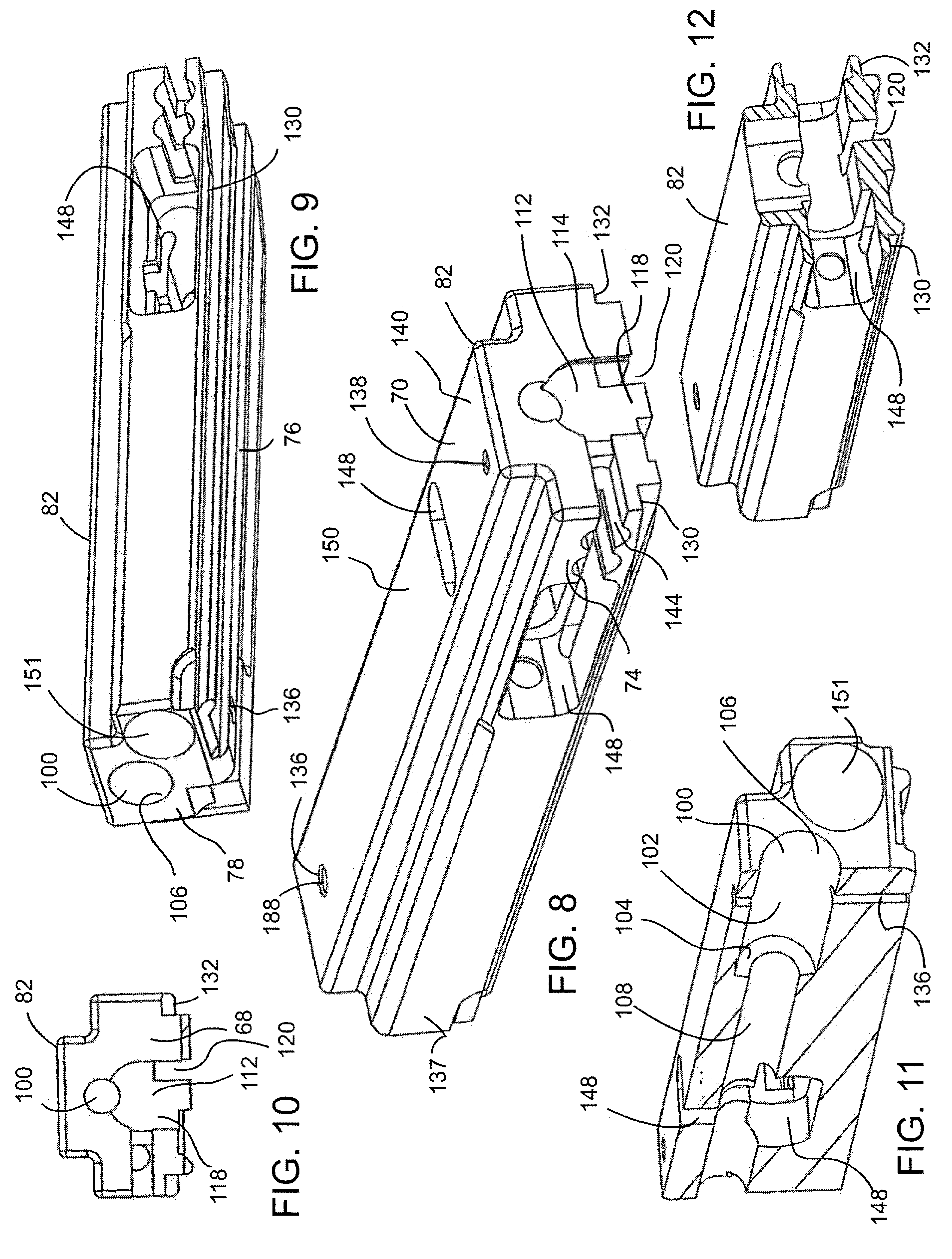
FIG. 8
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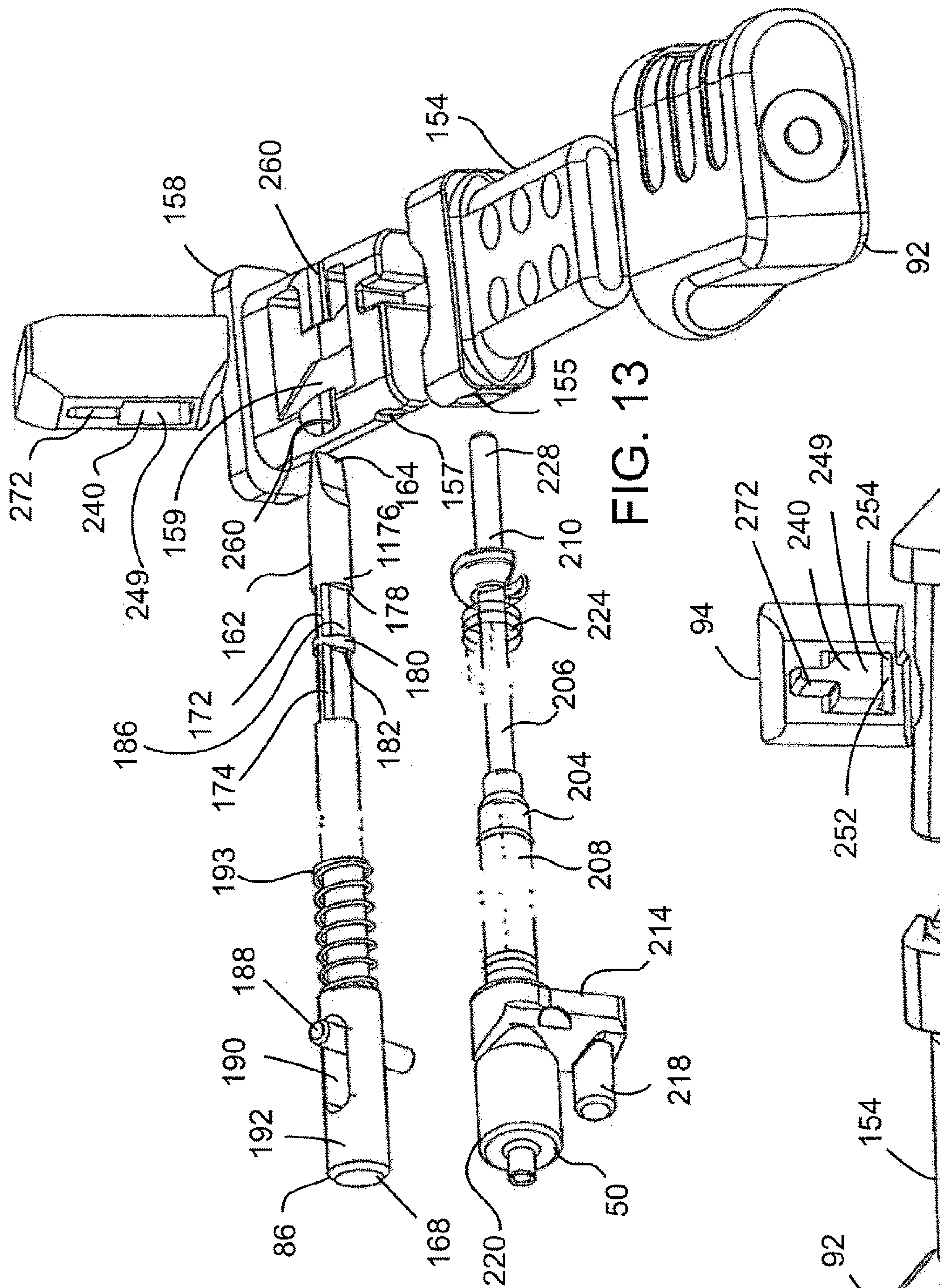
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FIG. 13

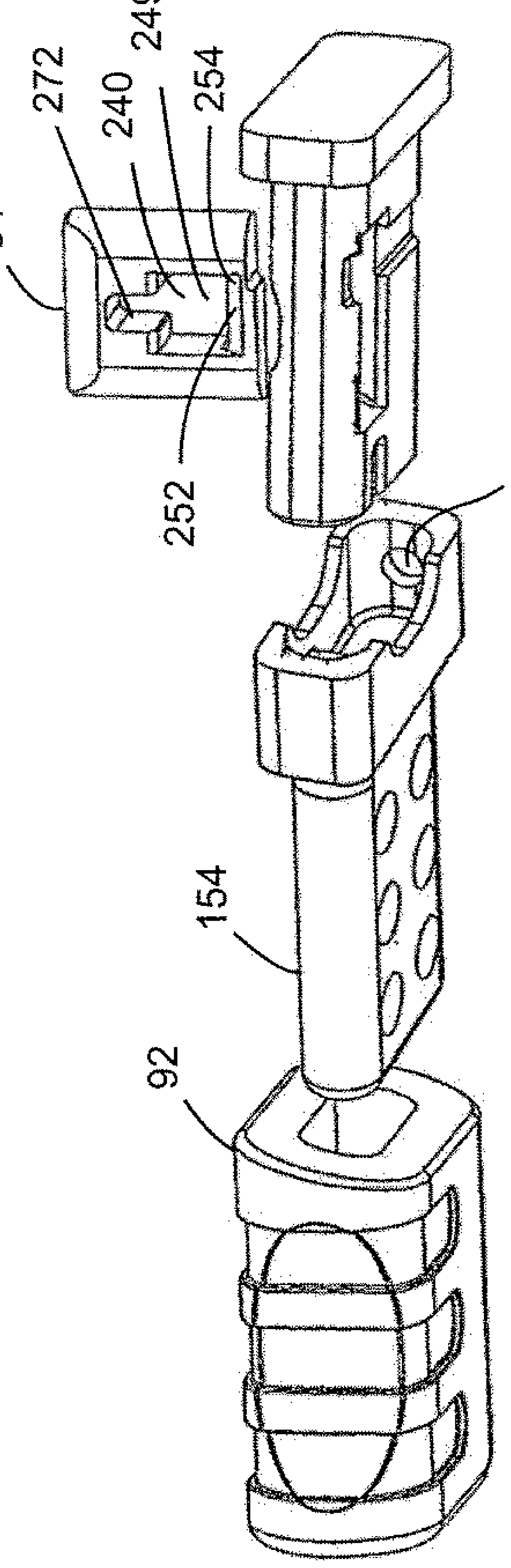
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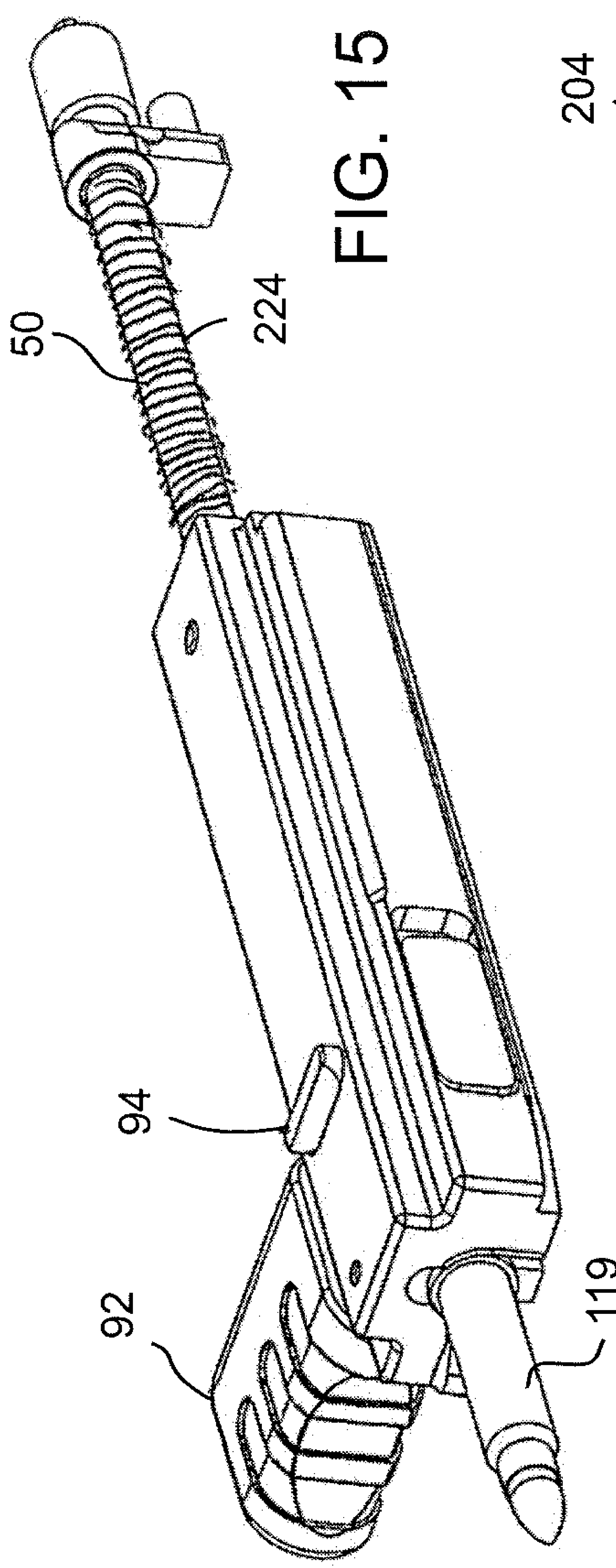
FIG. 14



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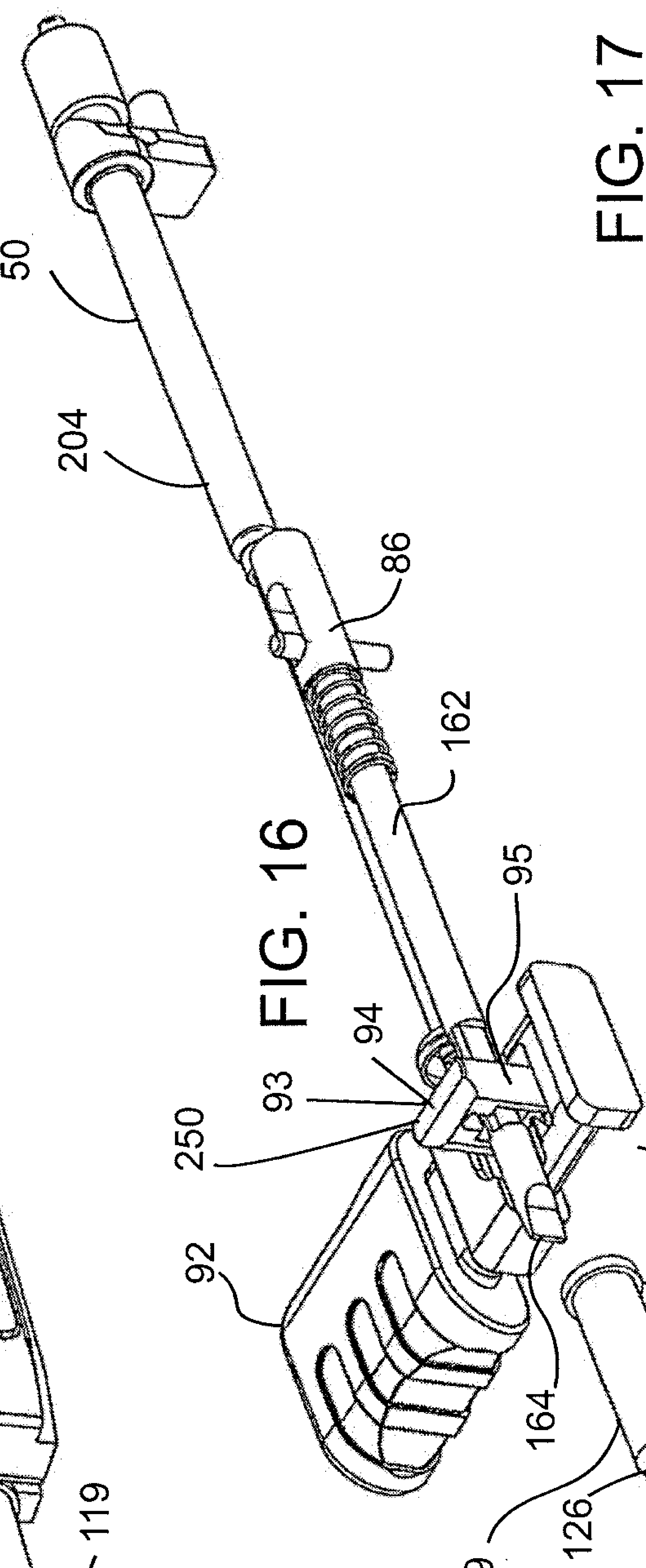

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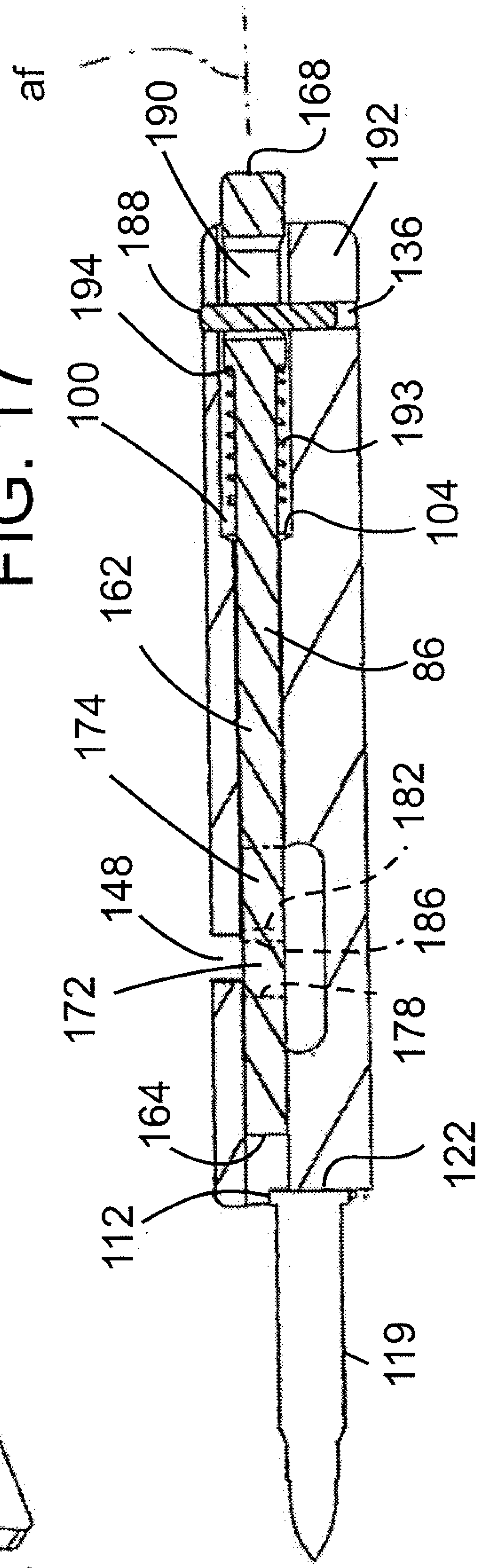

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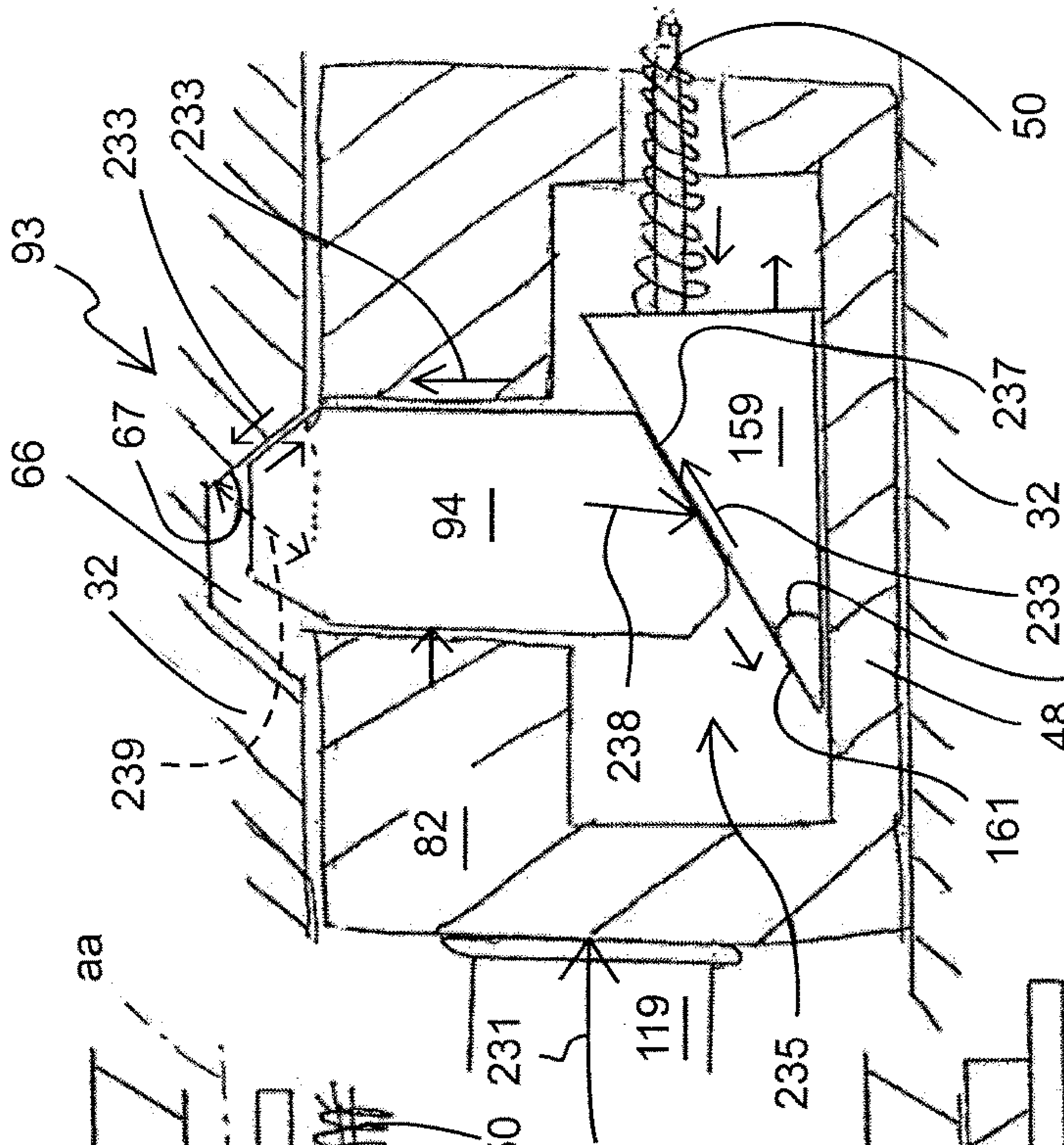

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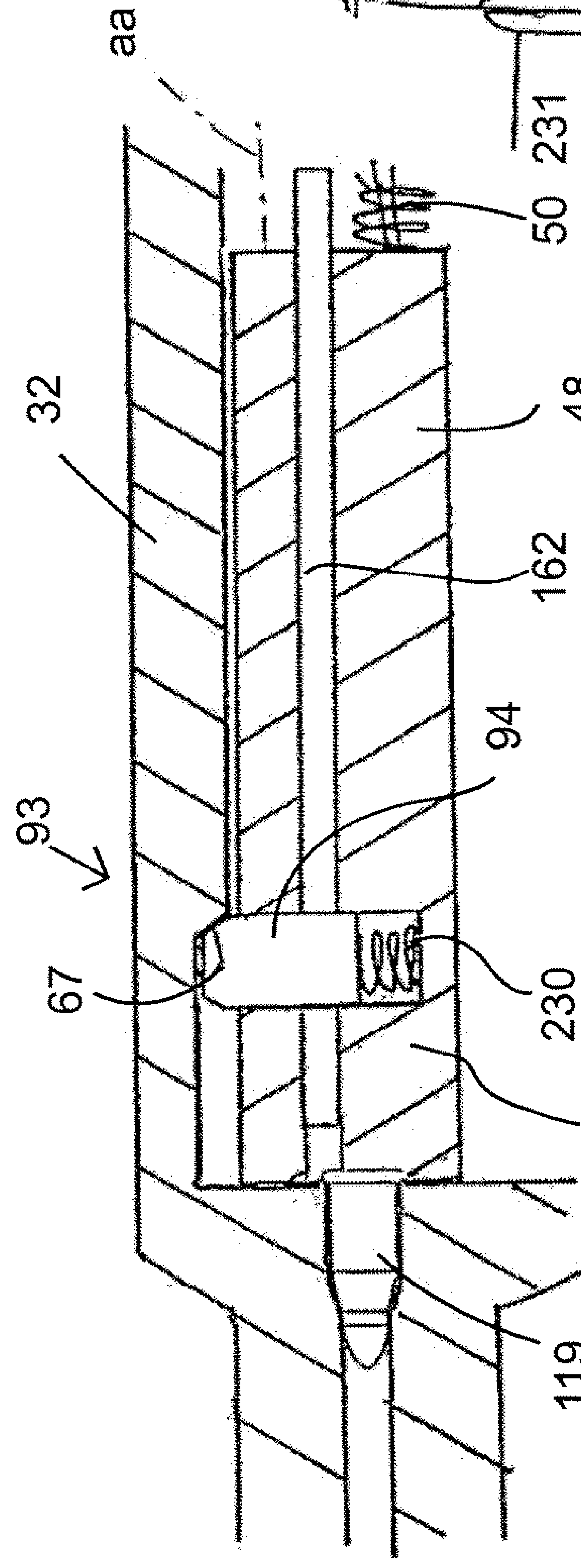

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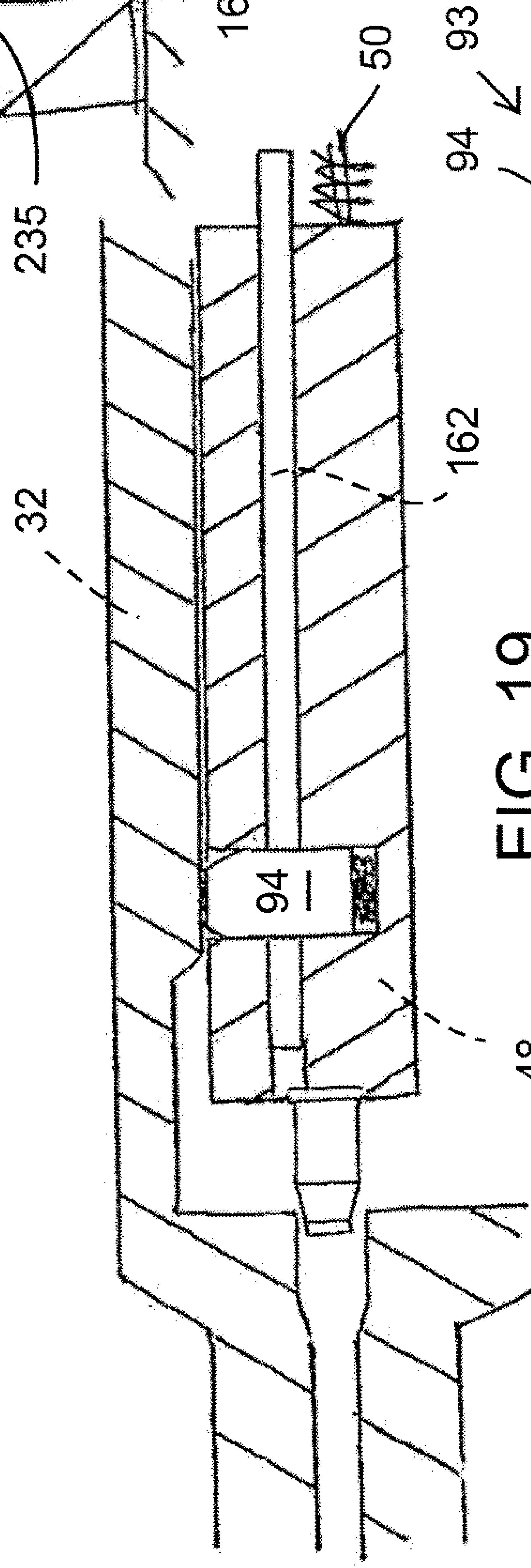

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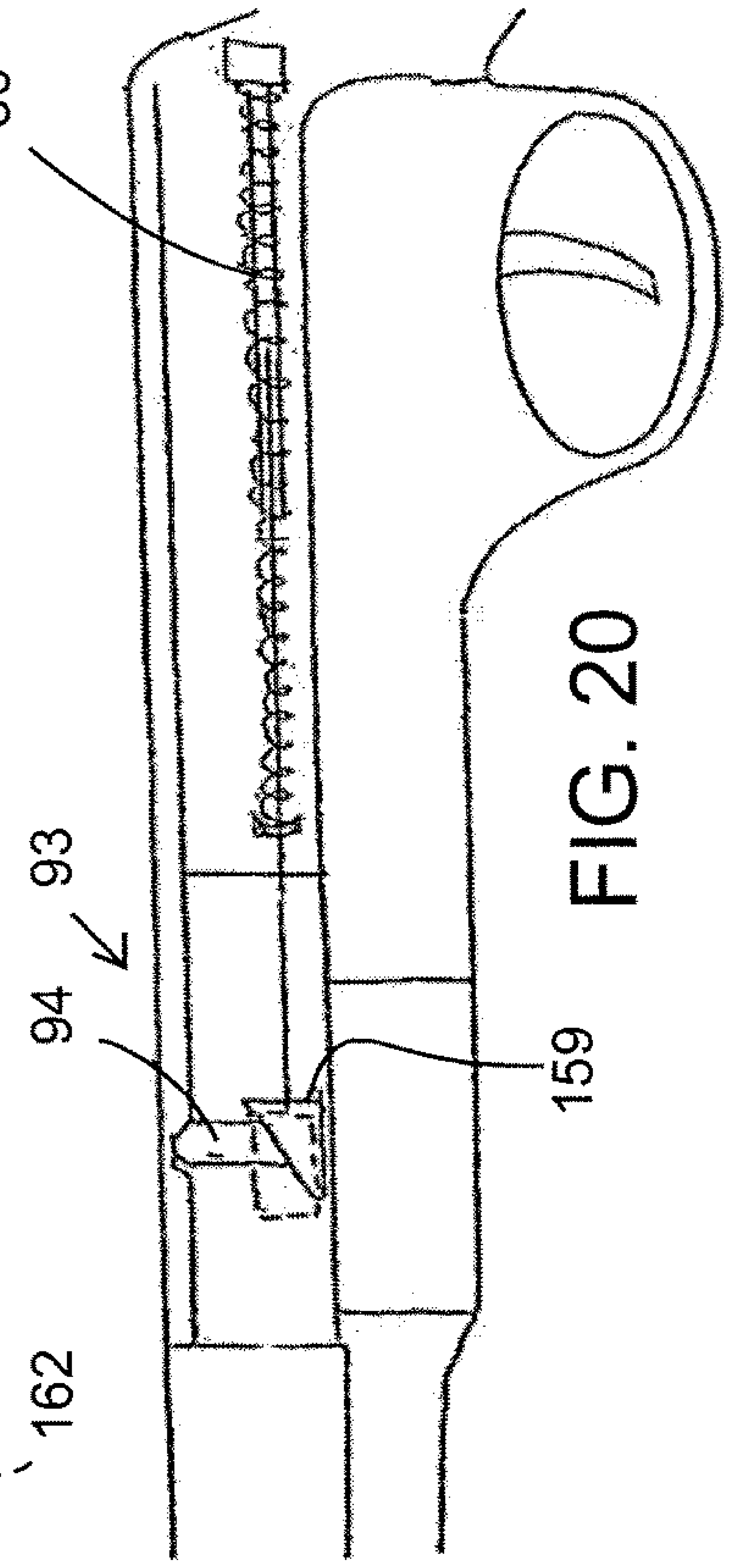

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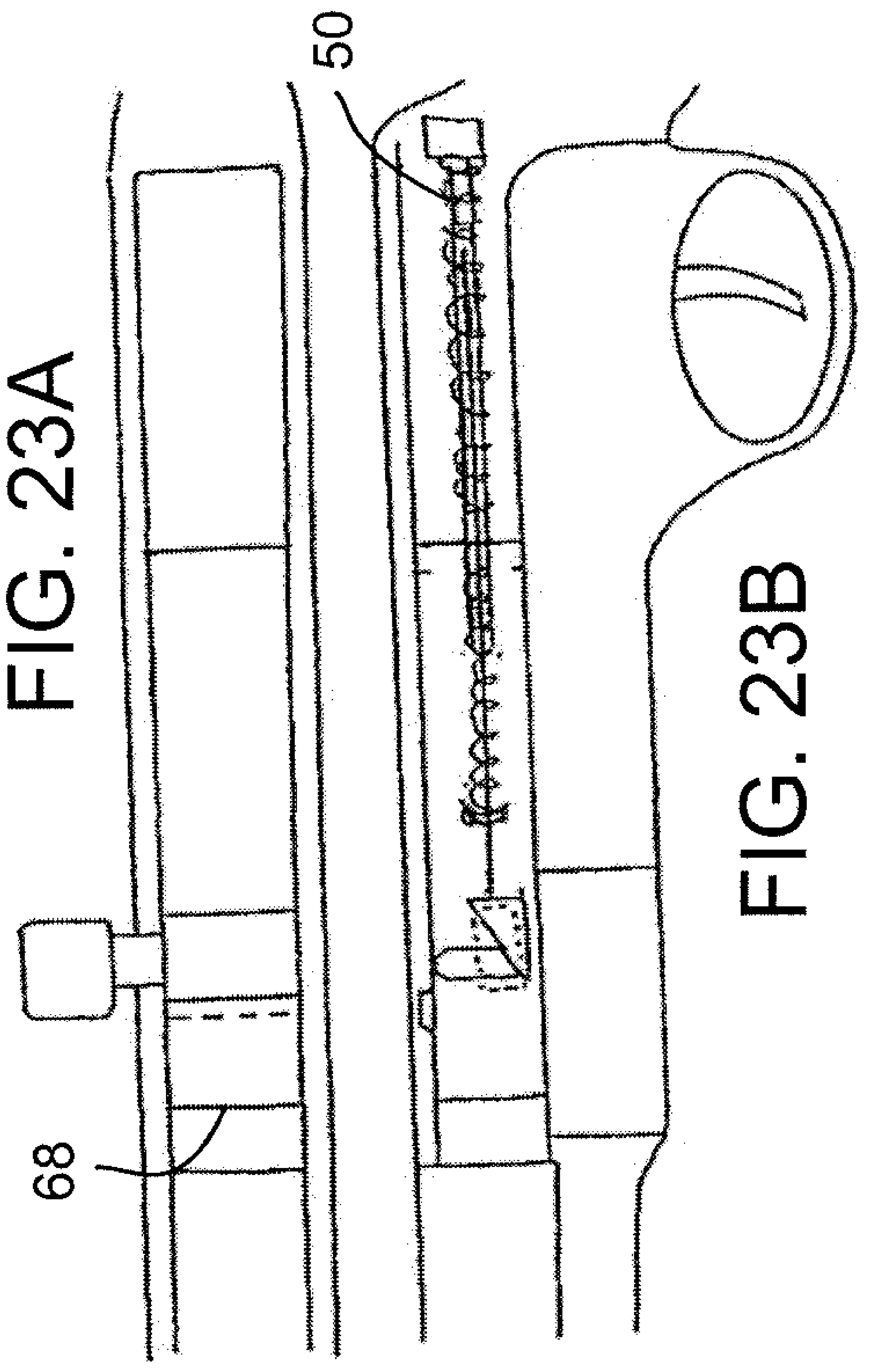


FIG. 23A

FIG. 23B

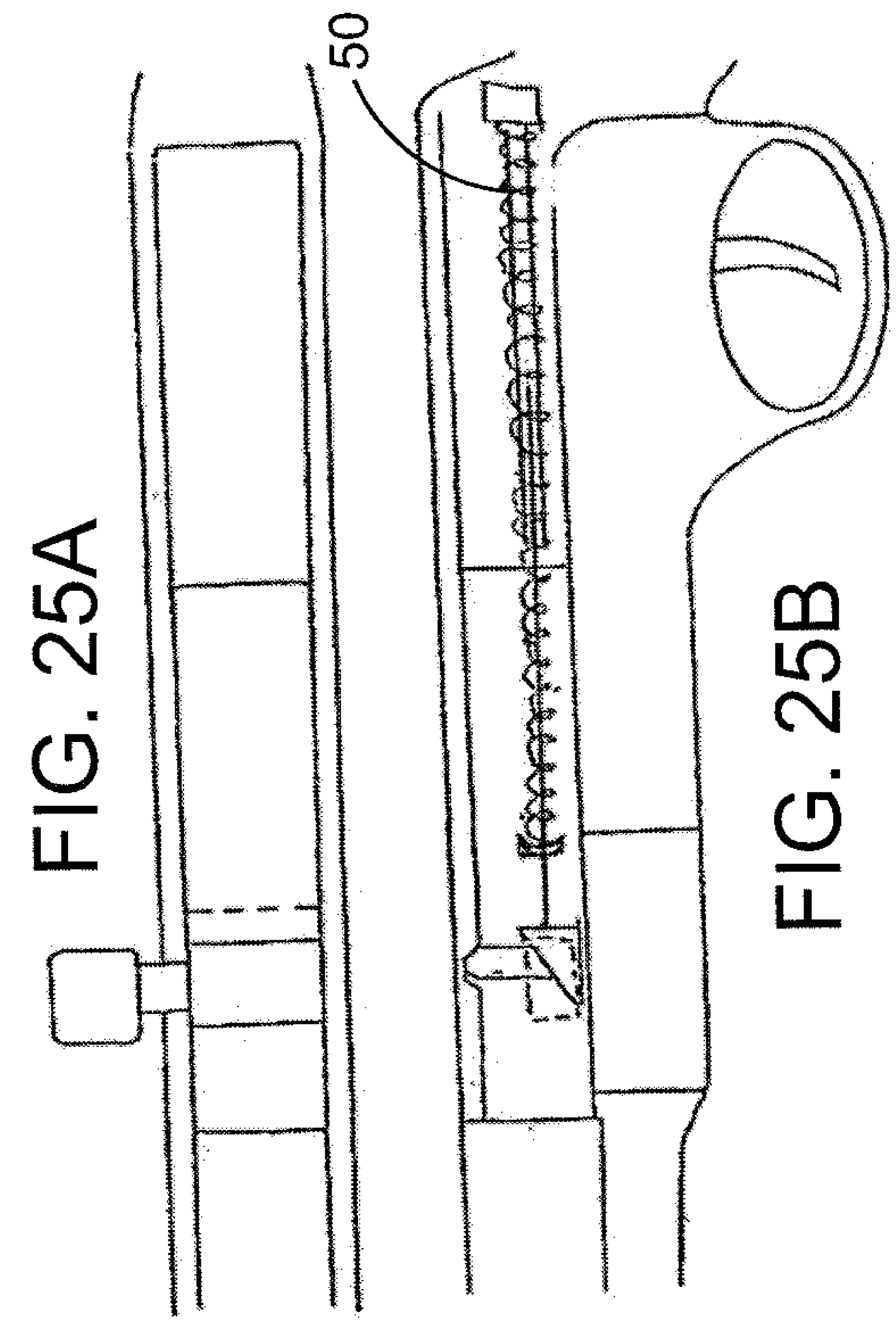


FIG. 25A

FIG. 25B

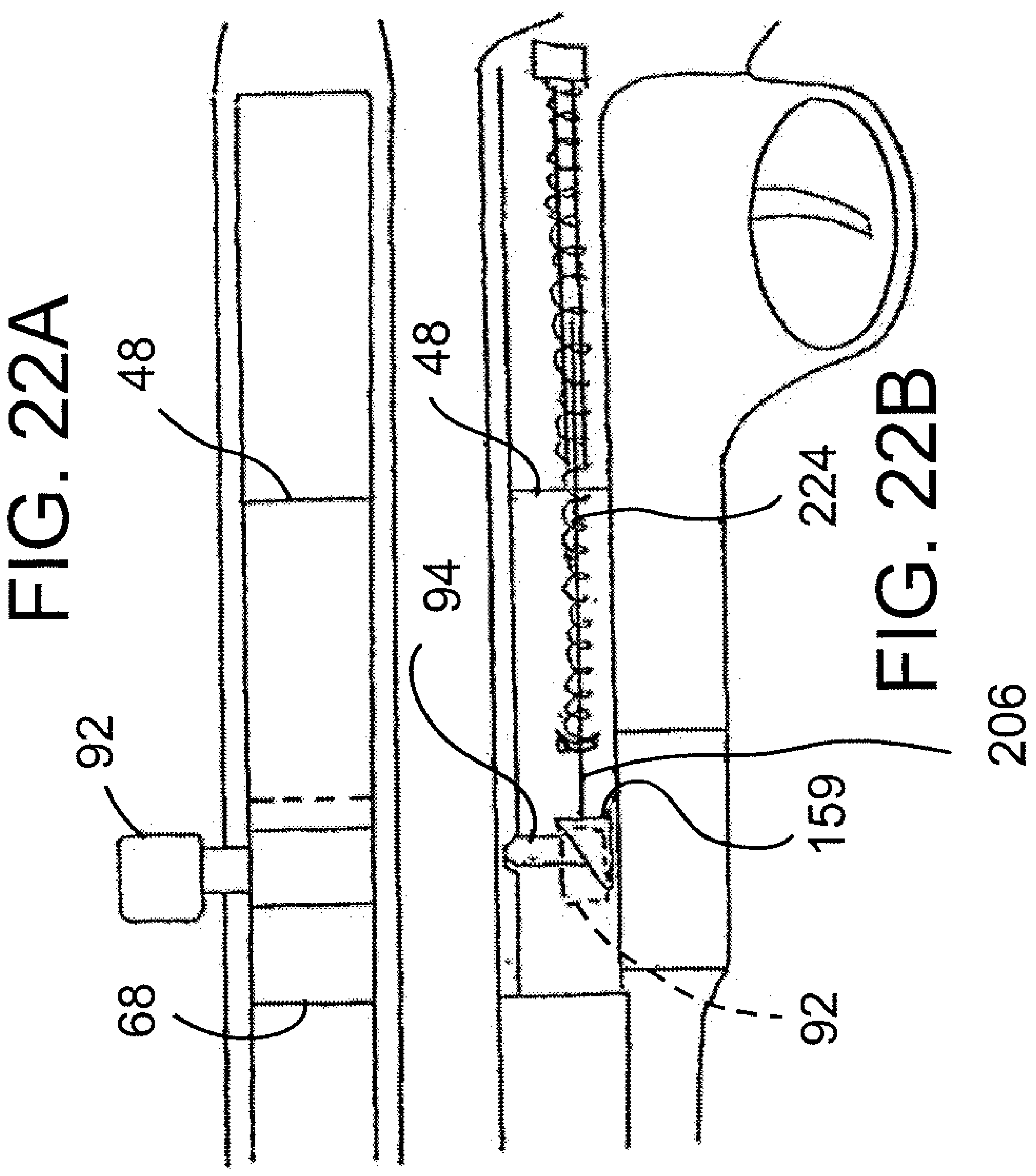


FIG. 22A

FIG. 22B

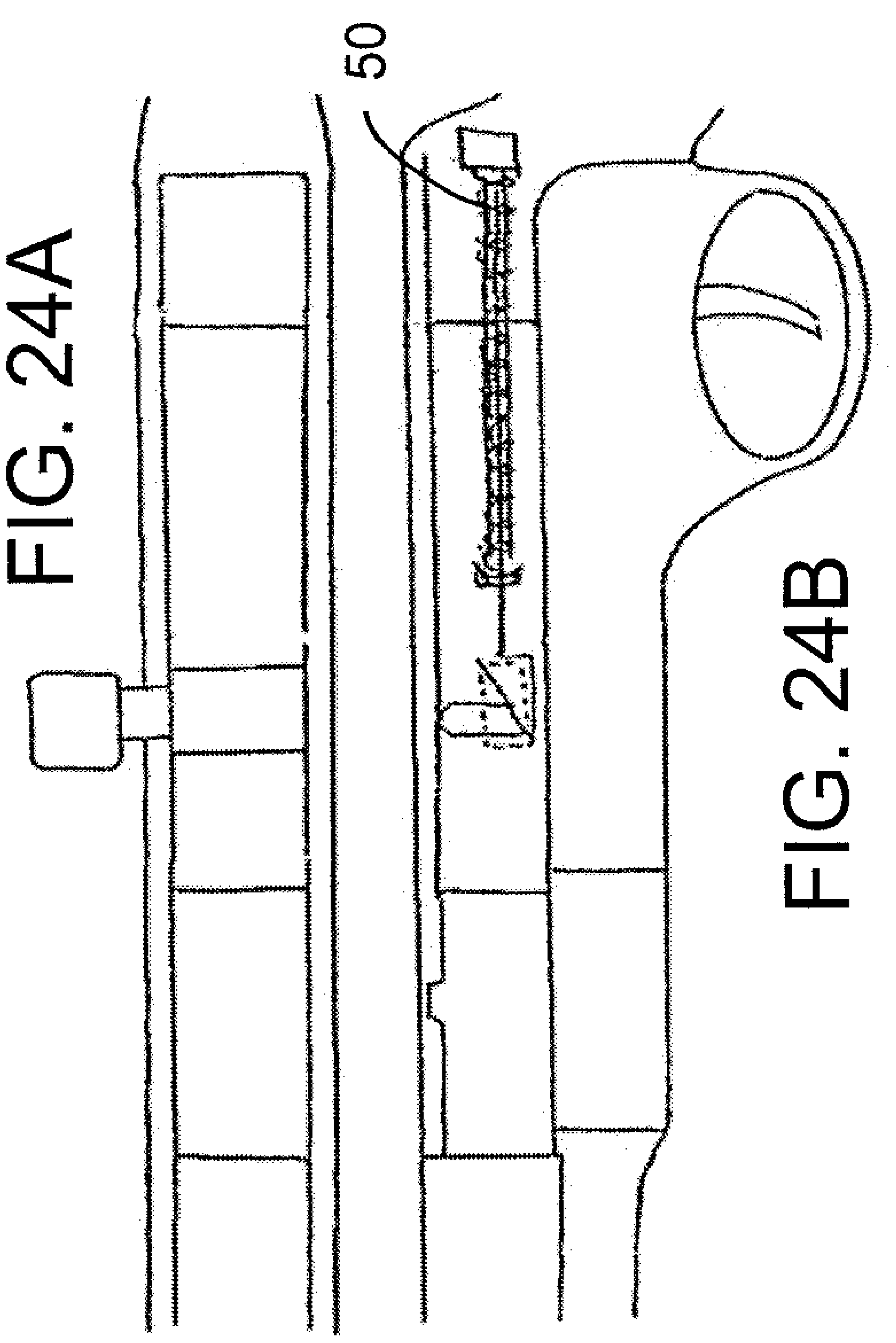
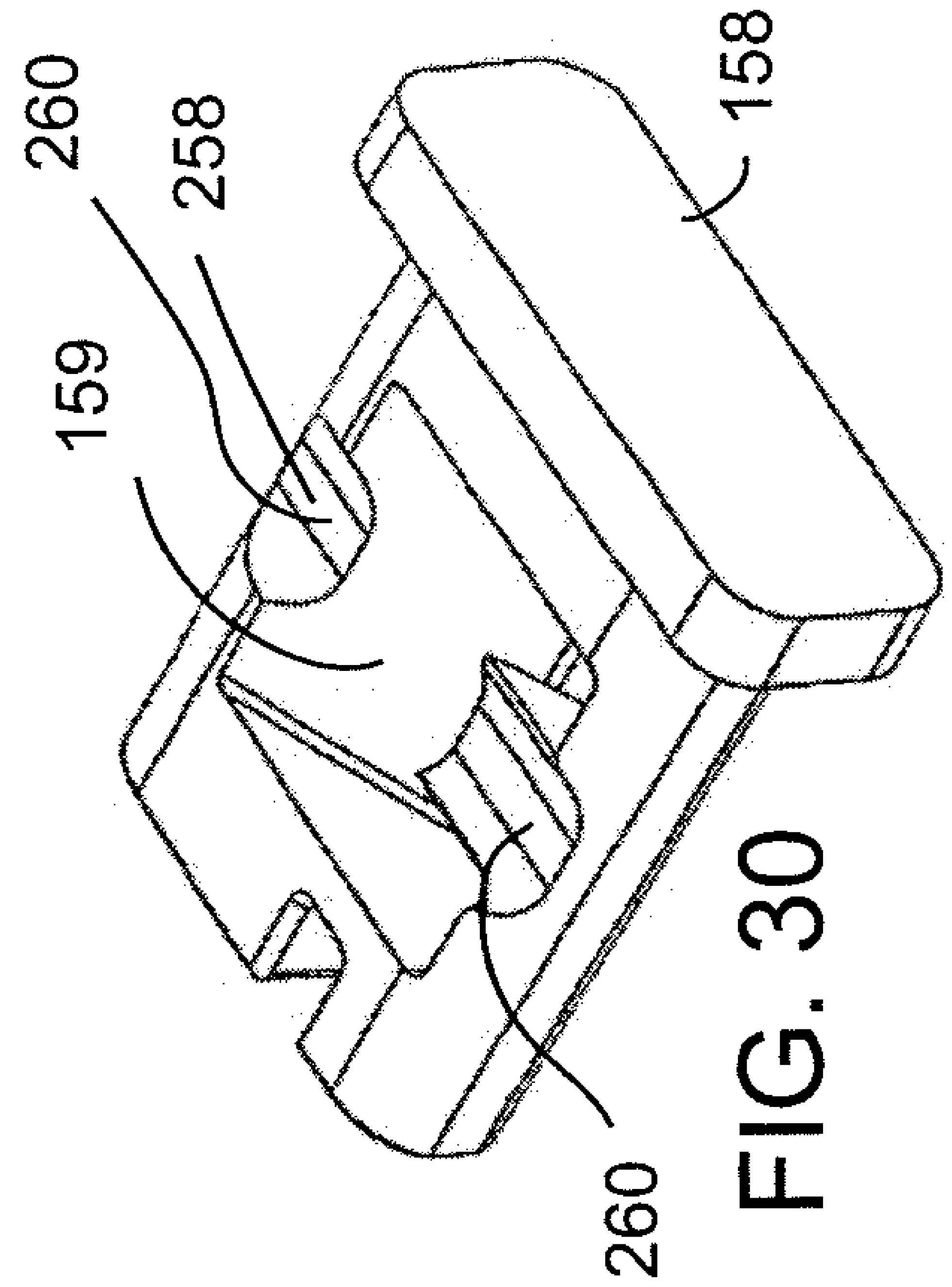


FIG. 24A

FIG. 24B



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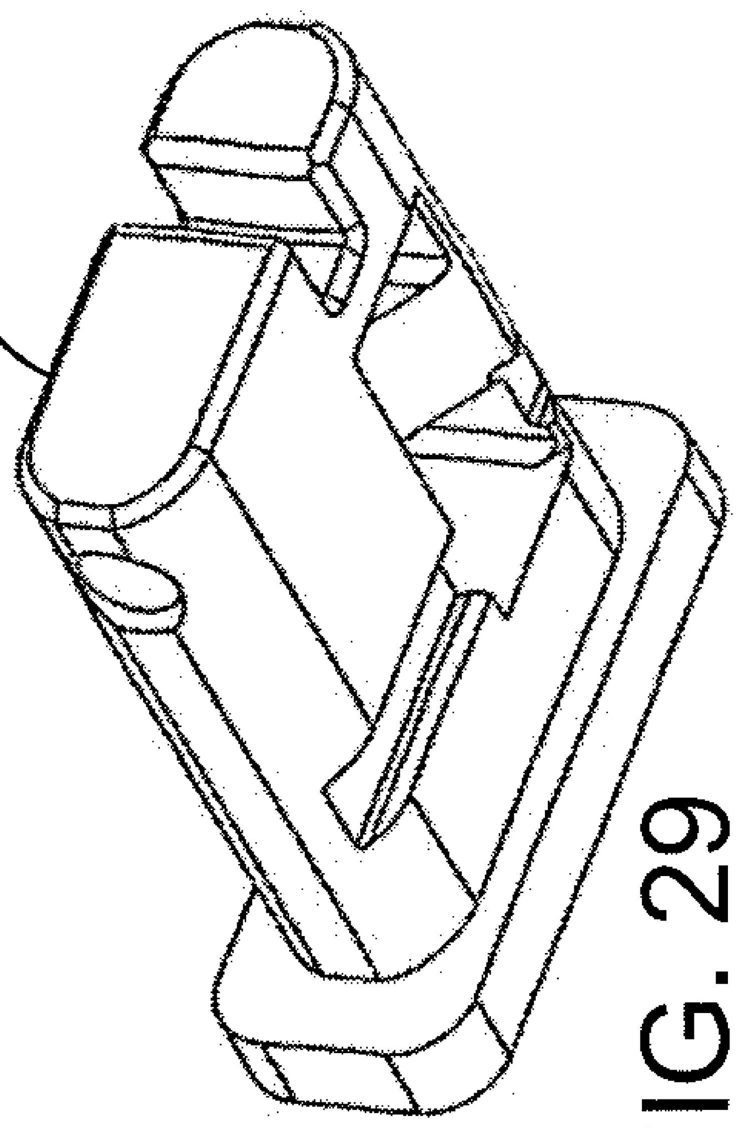

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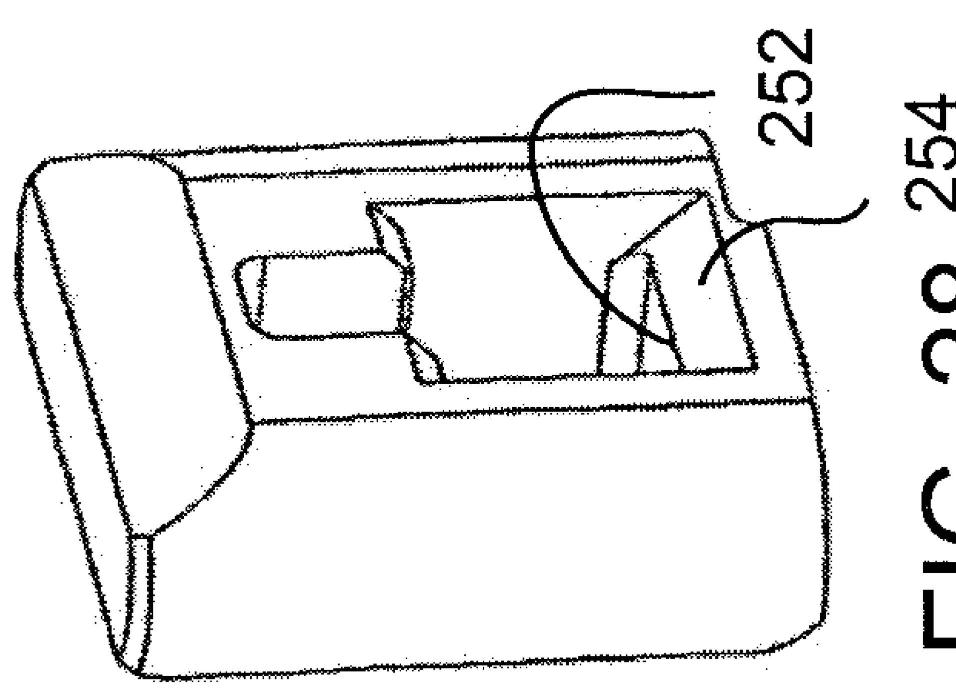


FIG. 28

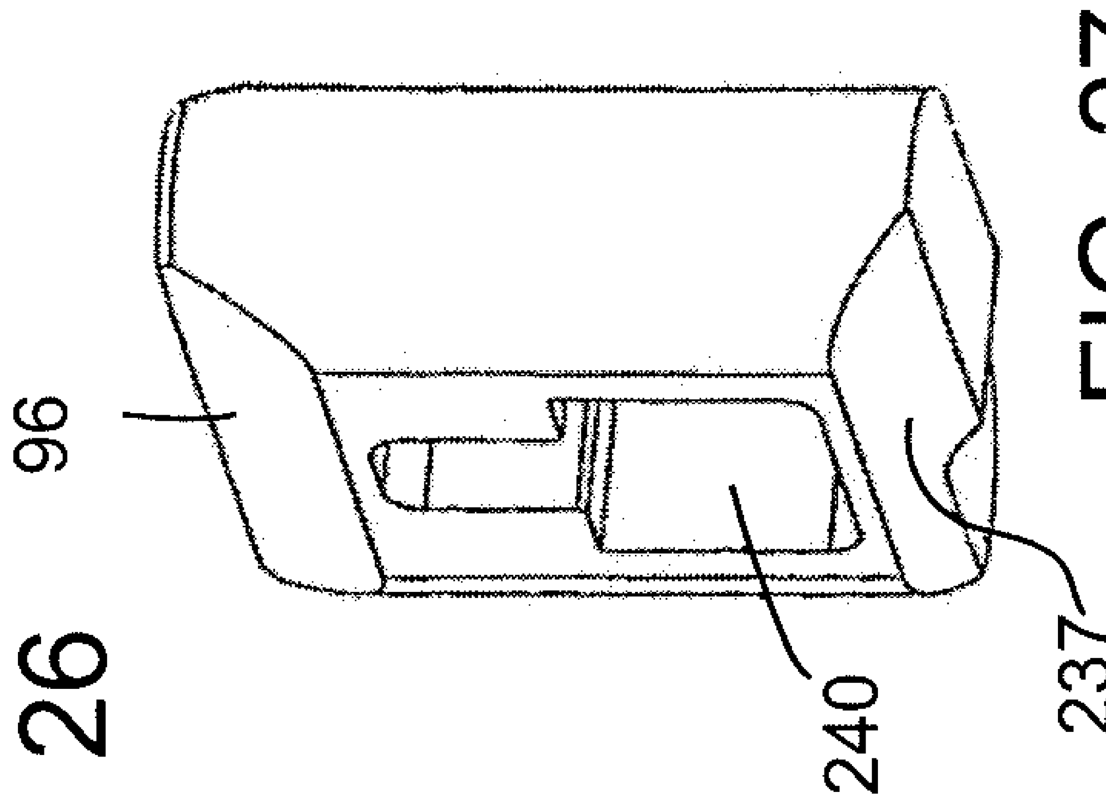
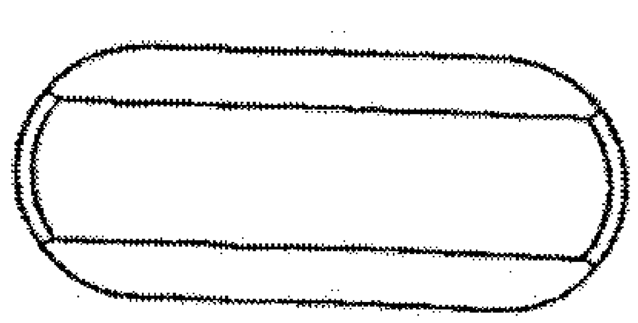


FIG. 27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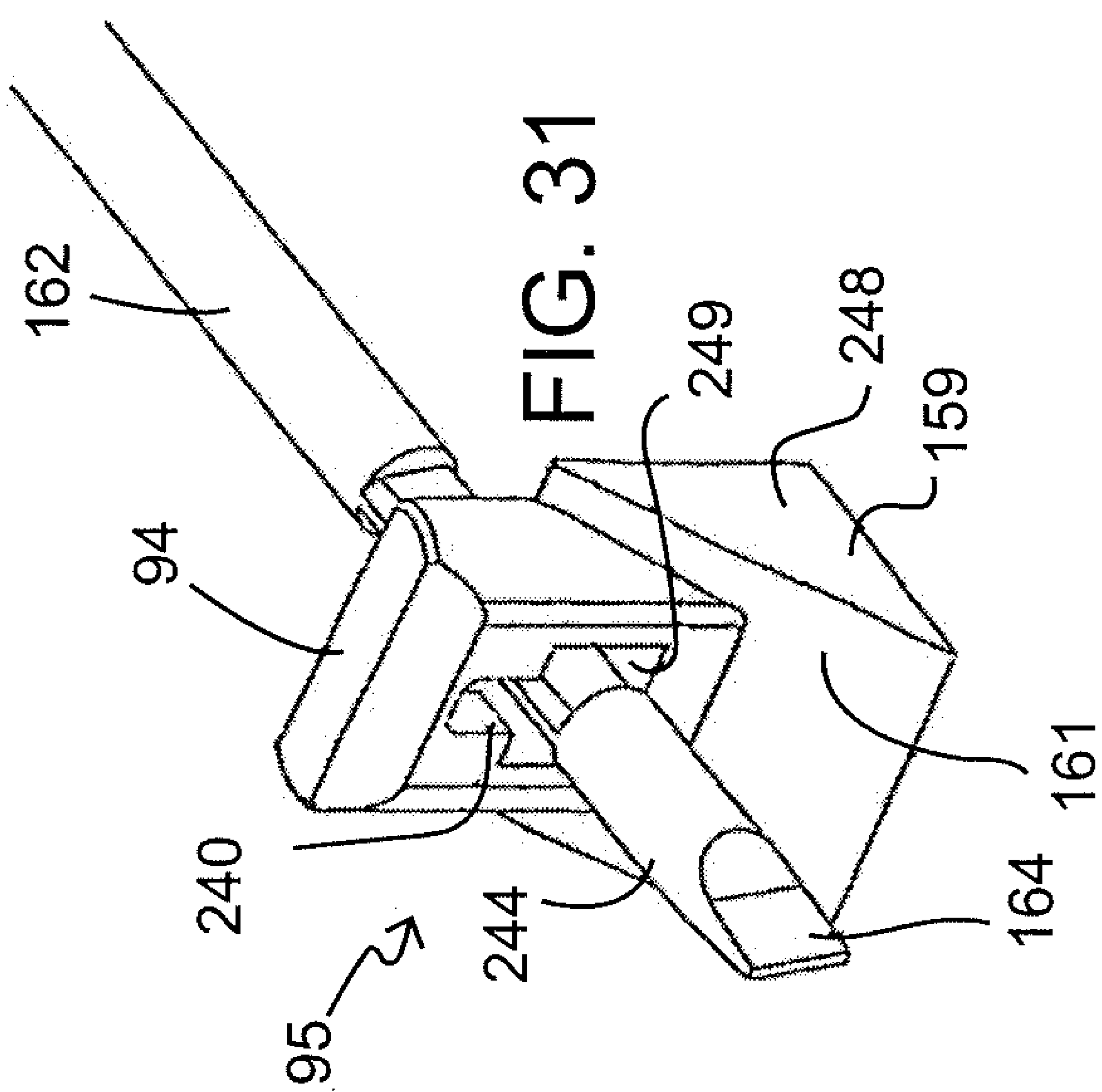

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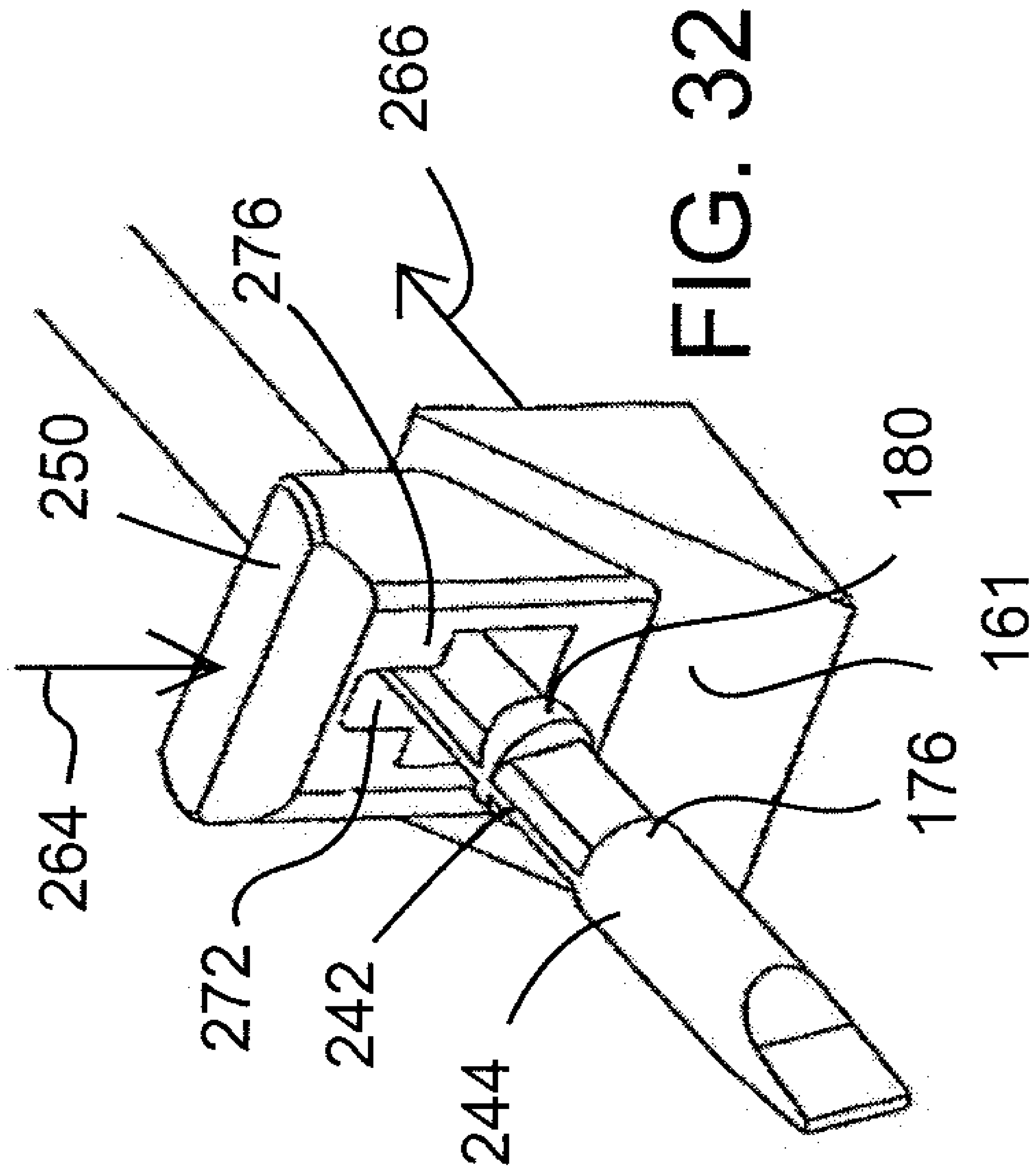

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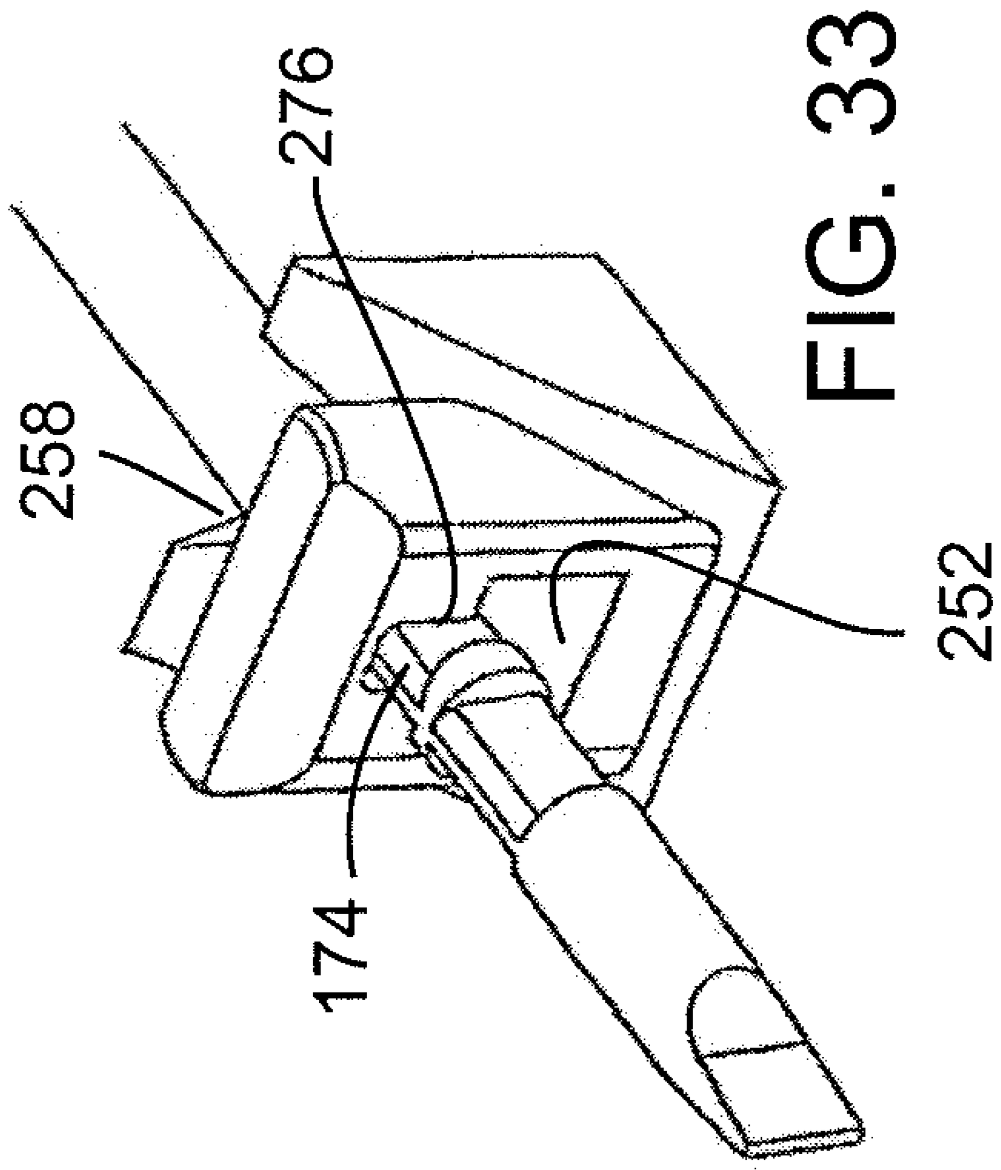

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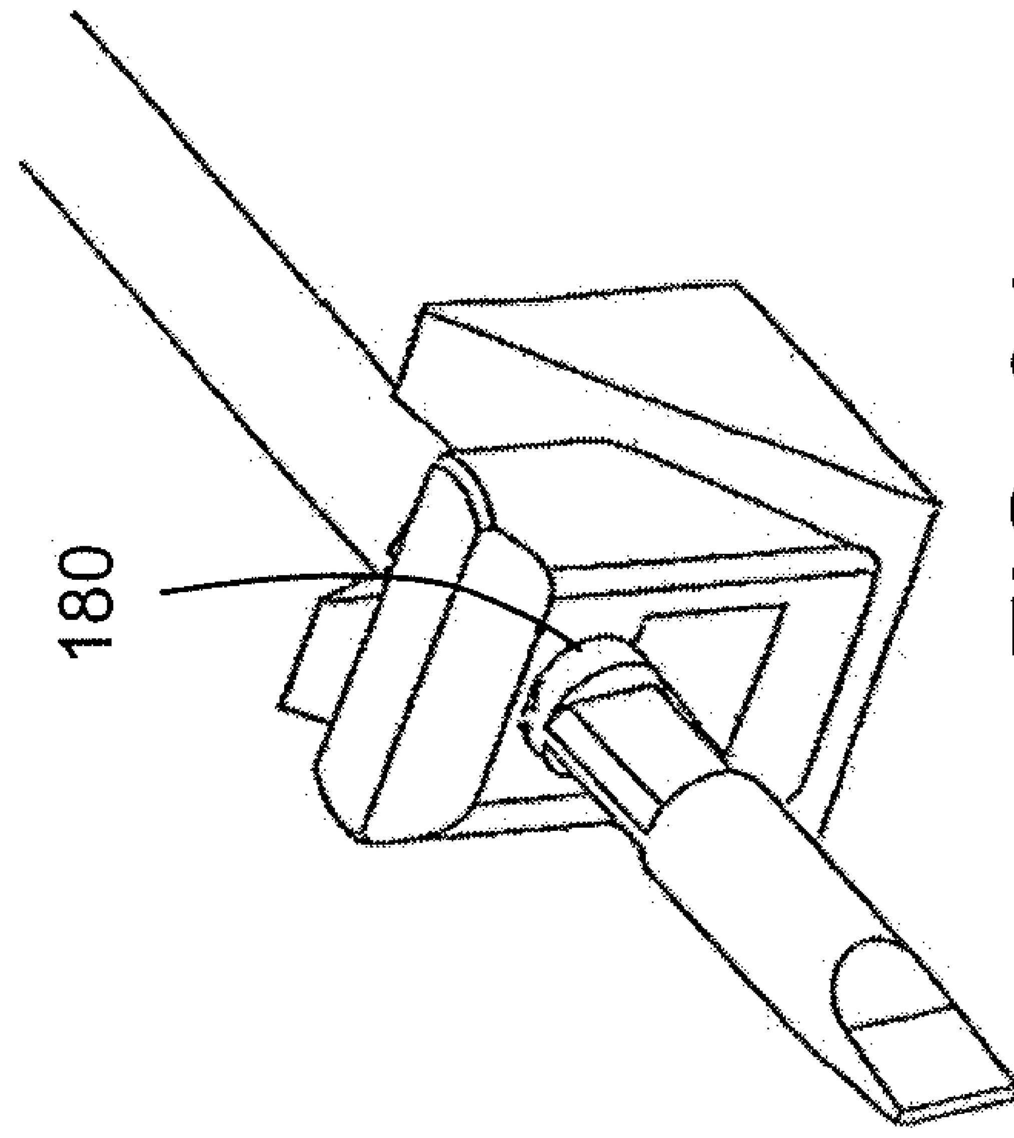


FIG. 34

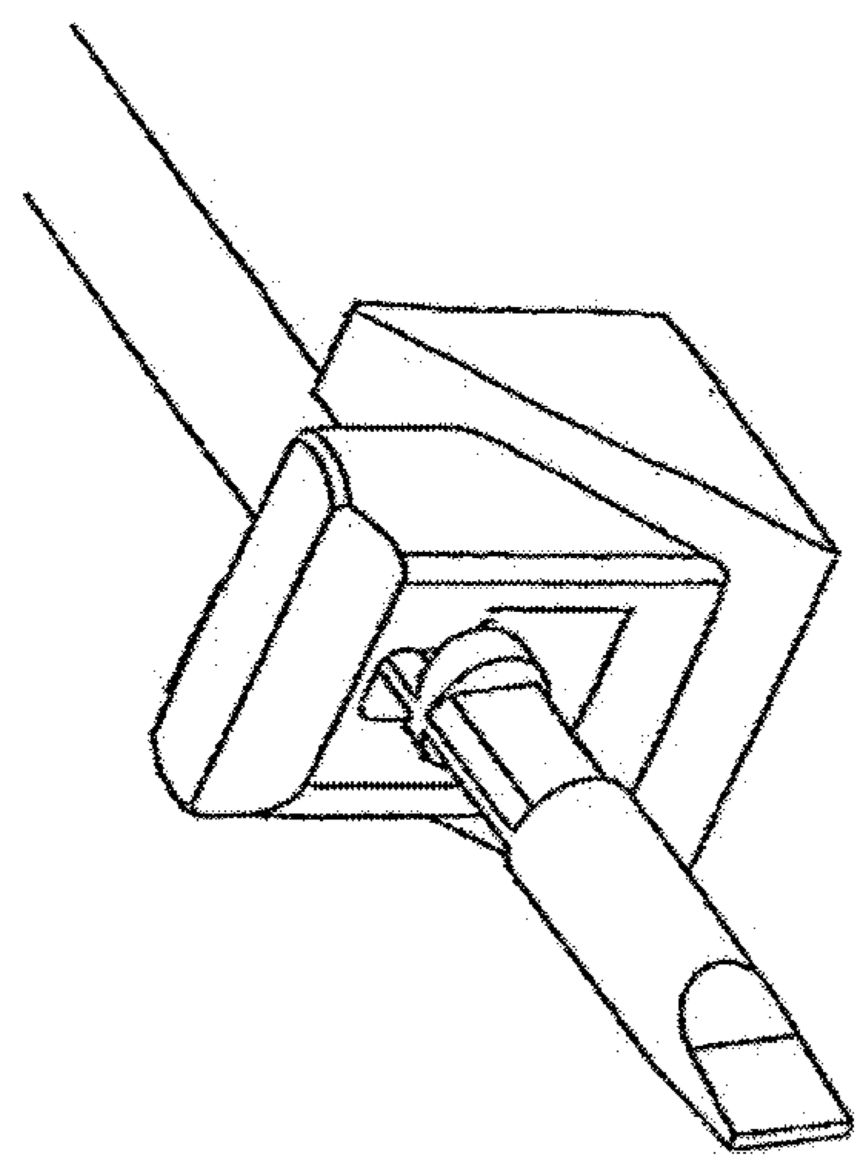


FIG. 35

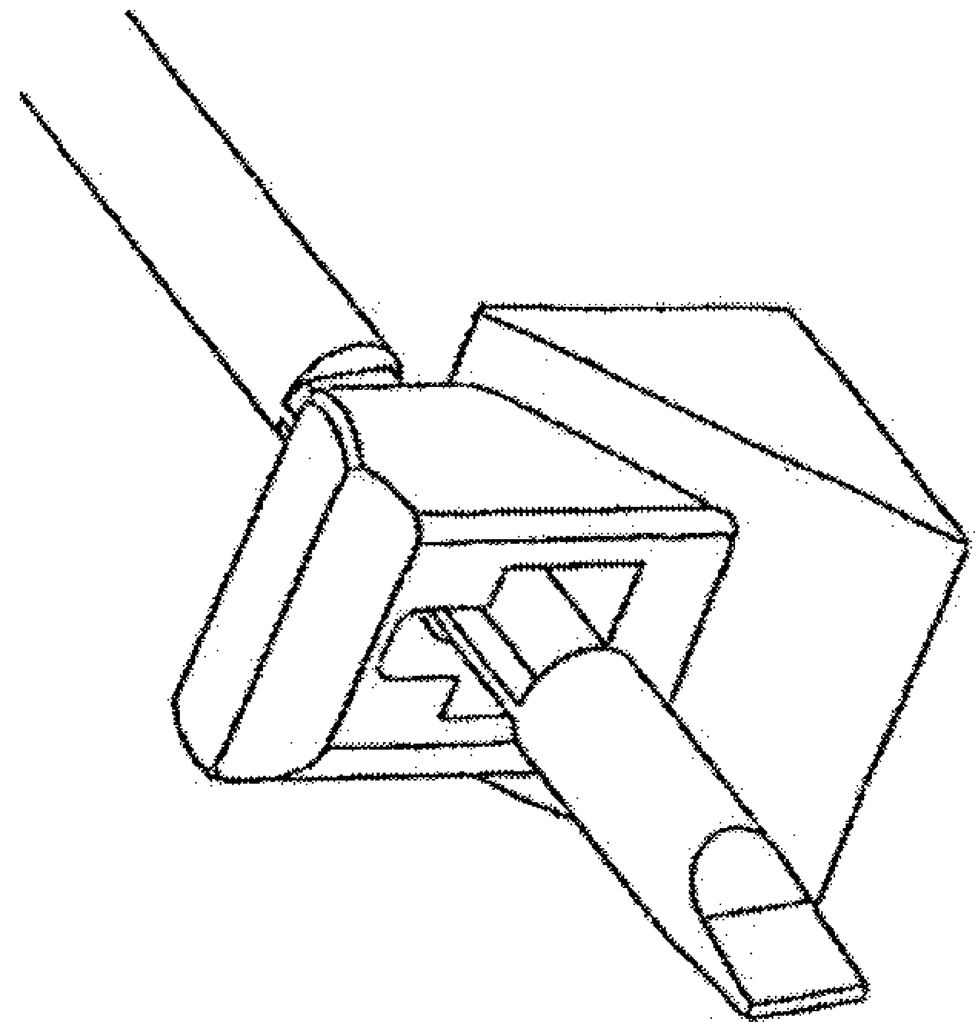


FIG. 36

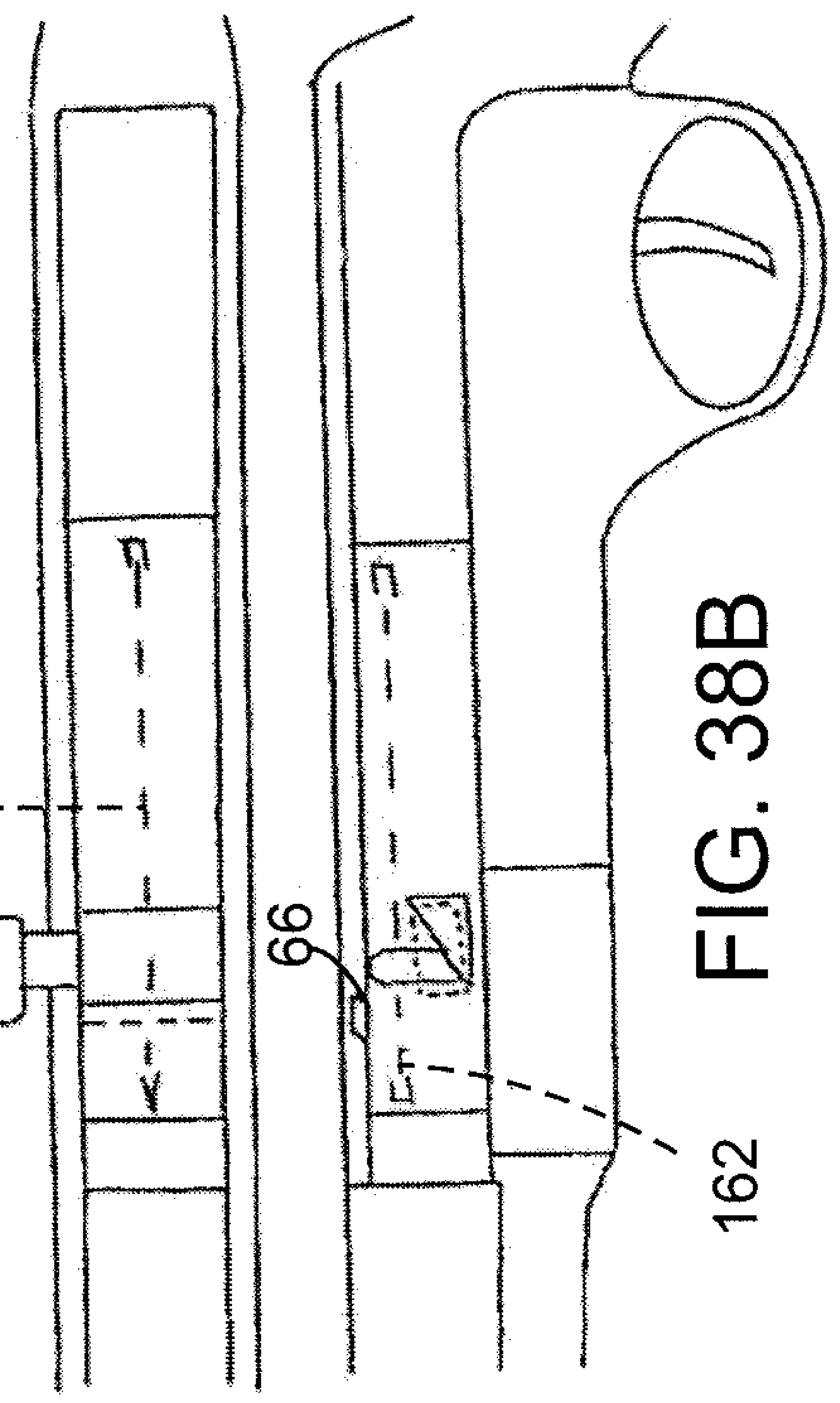


FIG. 38A

FIG. 38B

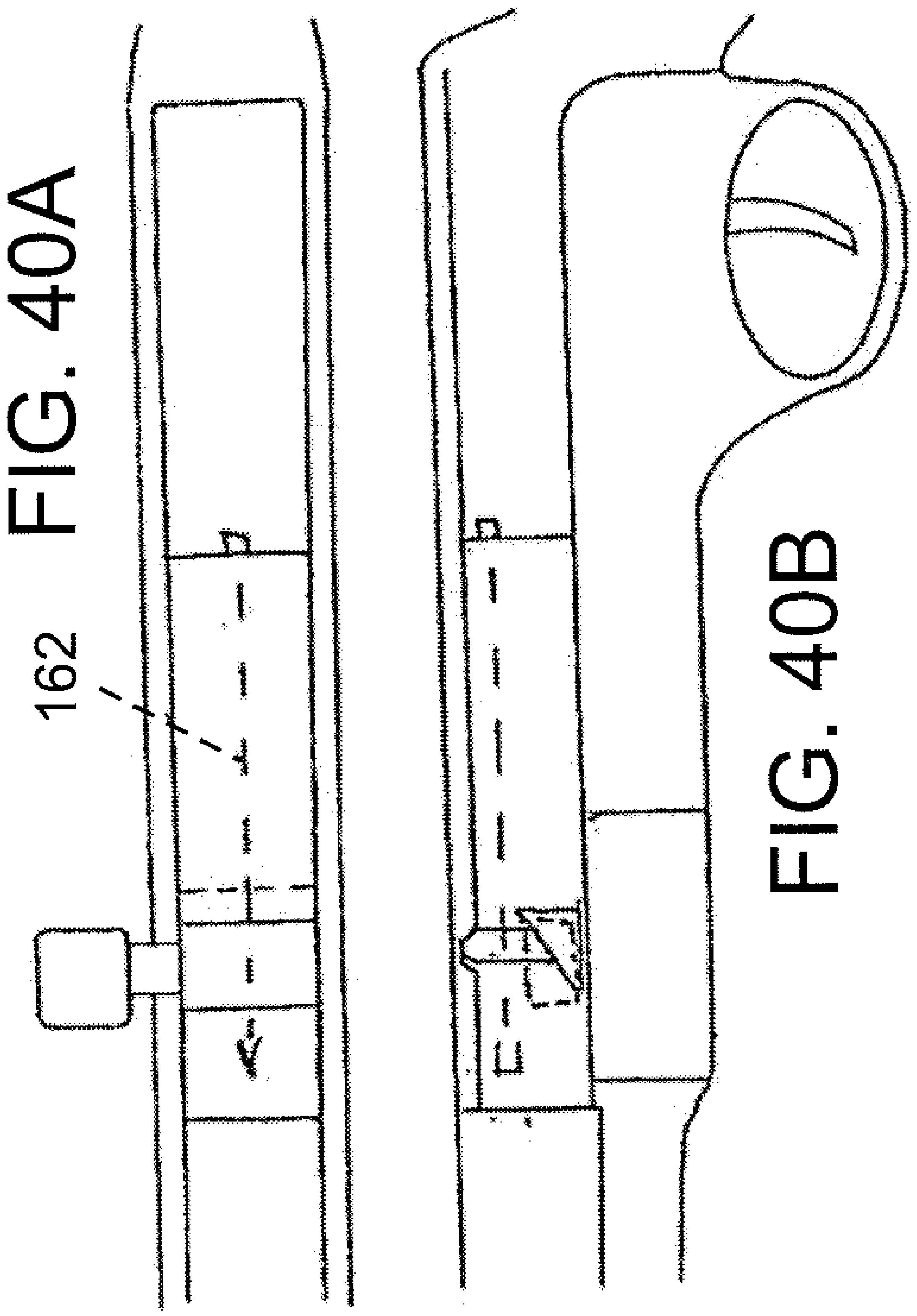


FIG. 40A

FIG. 40B

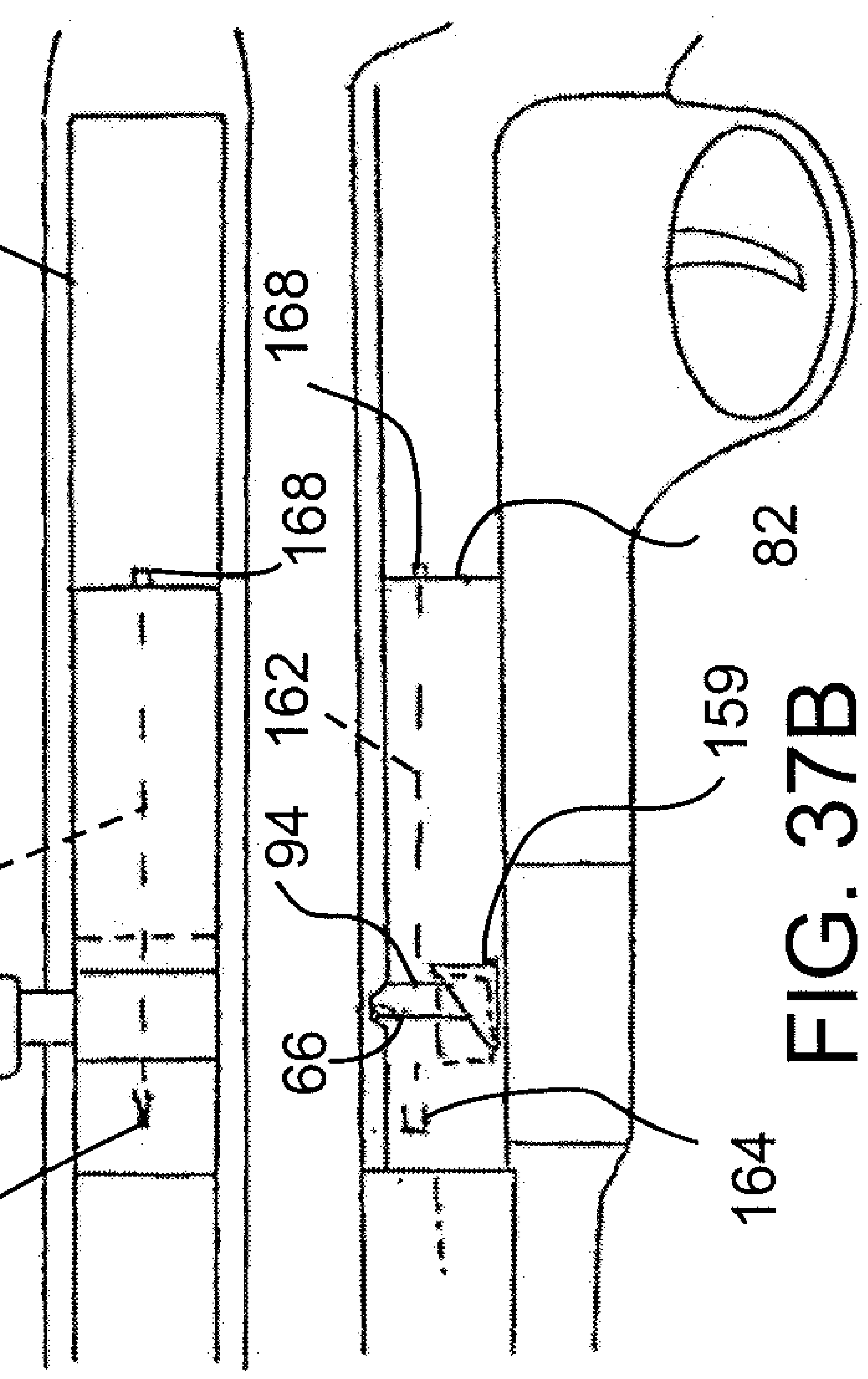


FIG. 37A

FIG. 37B

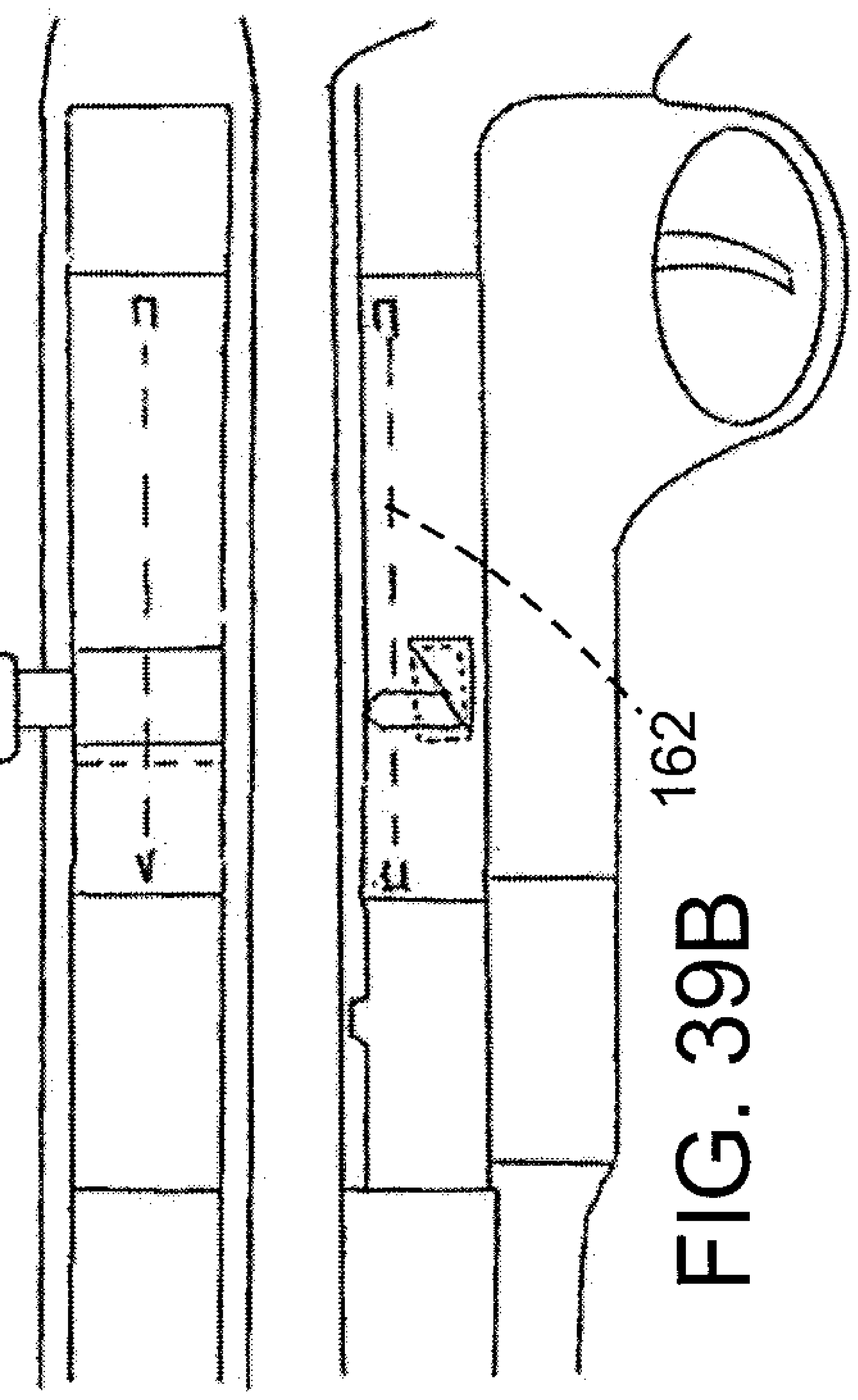


FIG. 39A

FIG. 39B

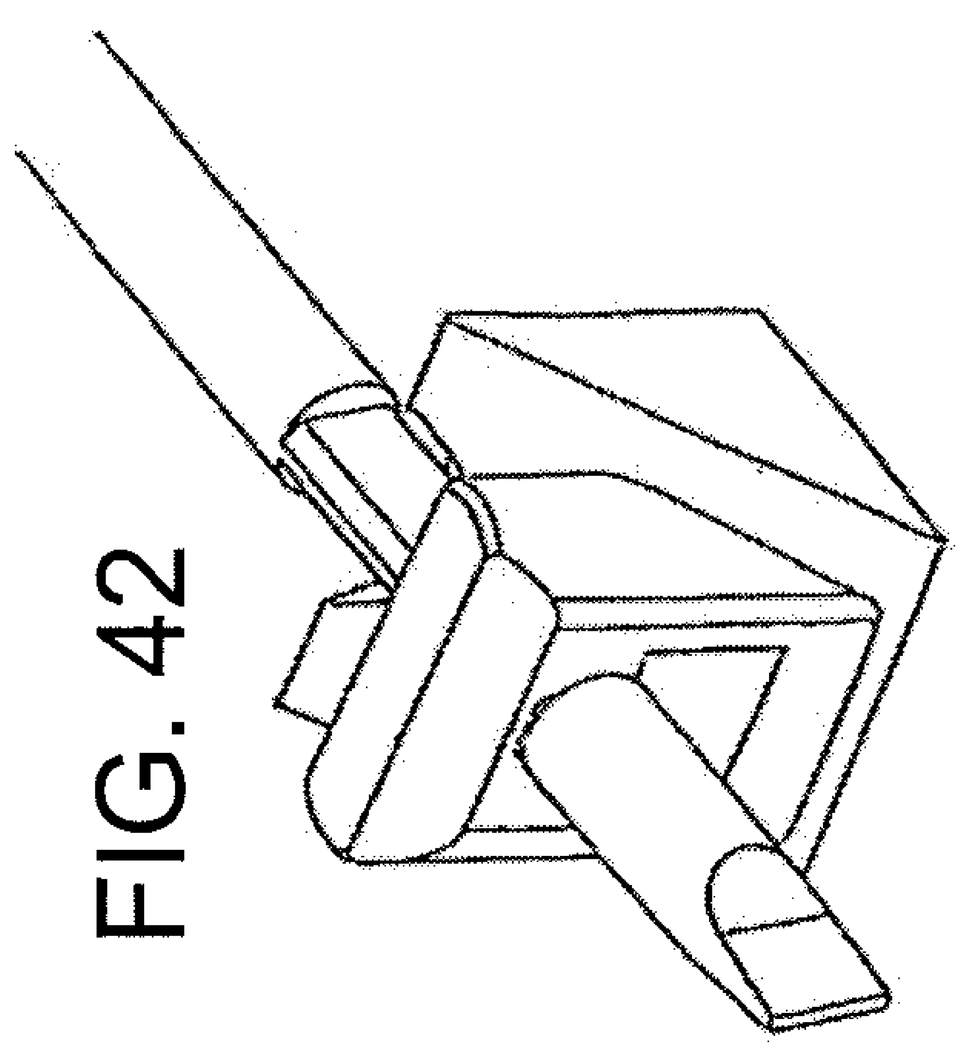


FIG. 42

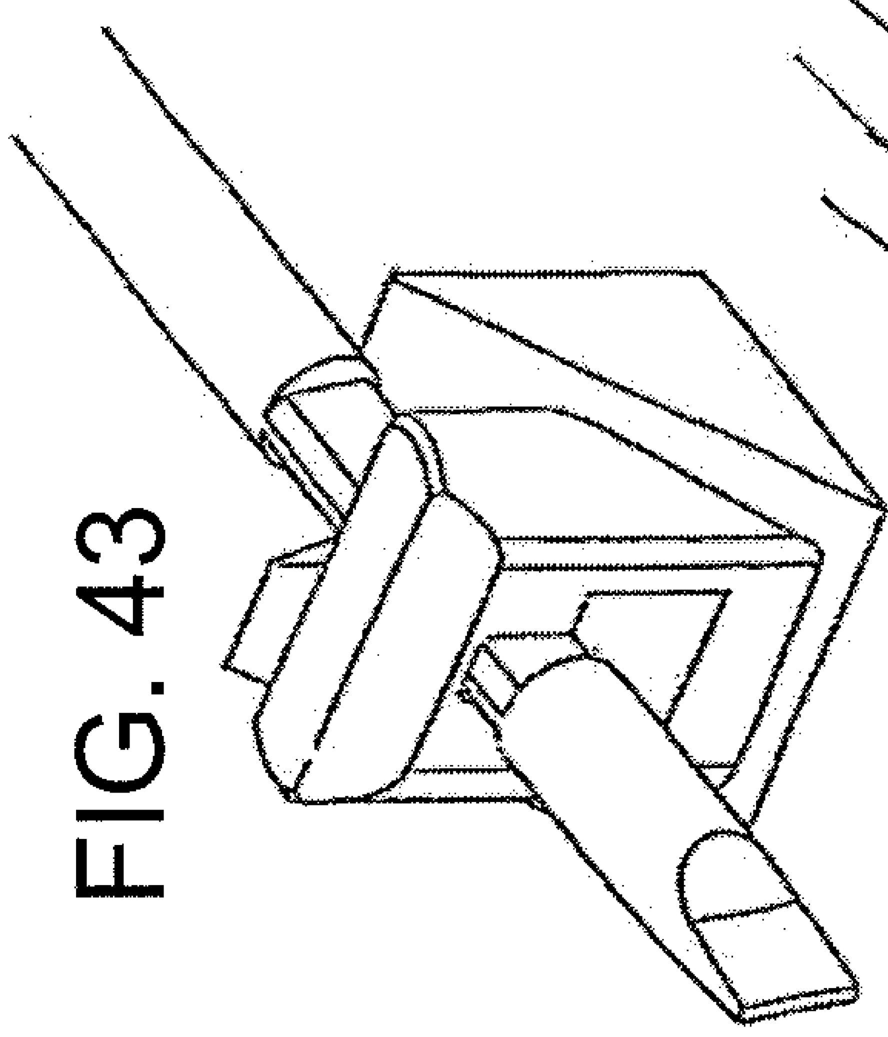


FIG. 43

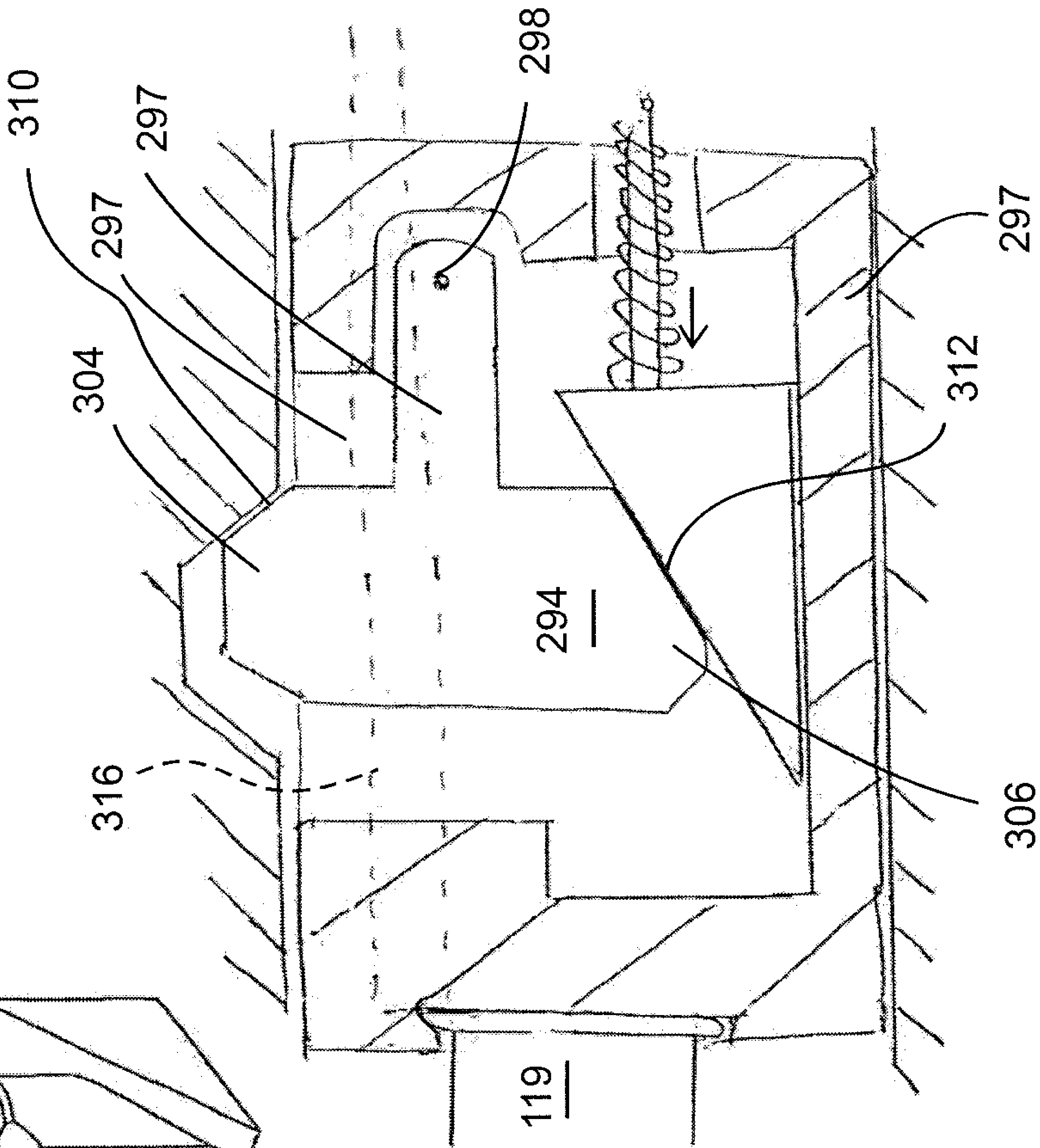


FIG. 44

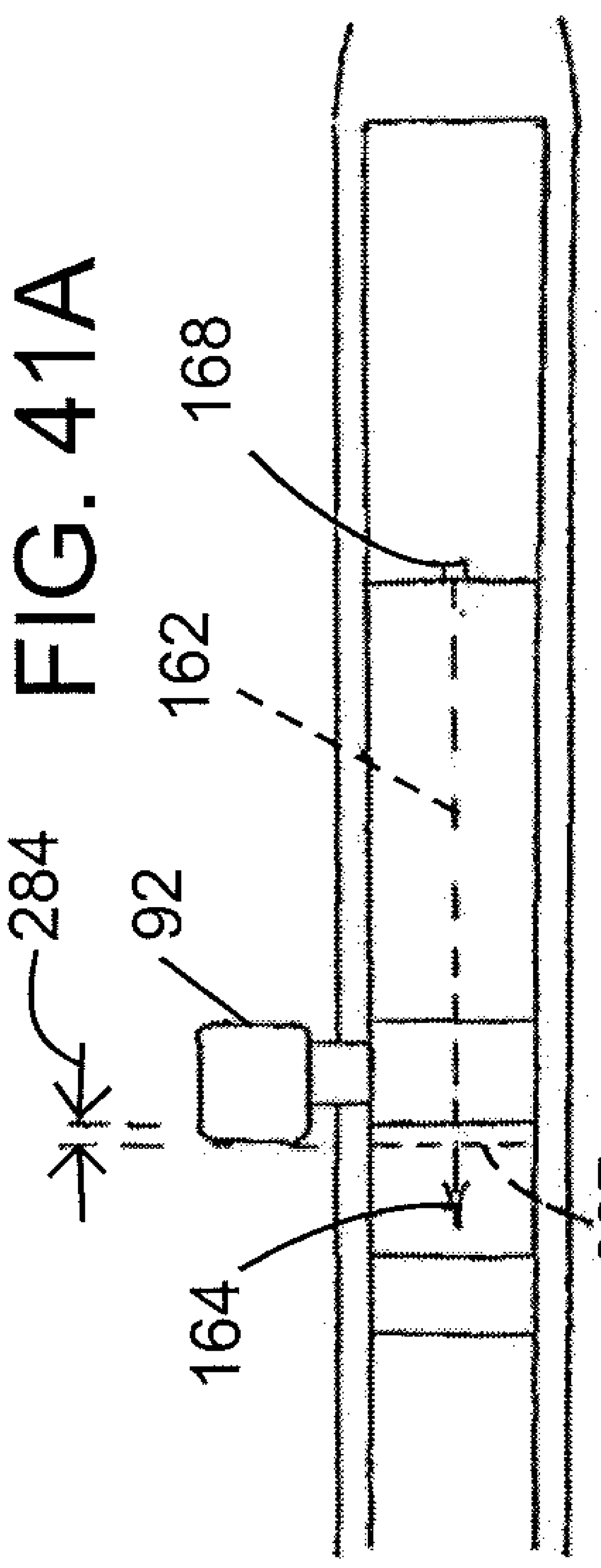


FIG. 41A

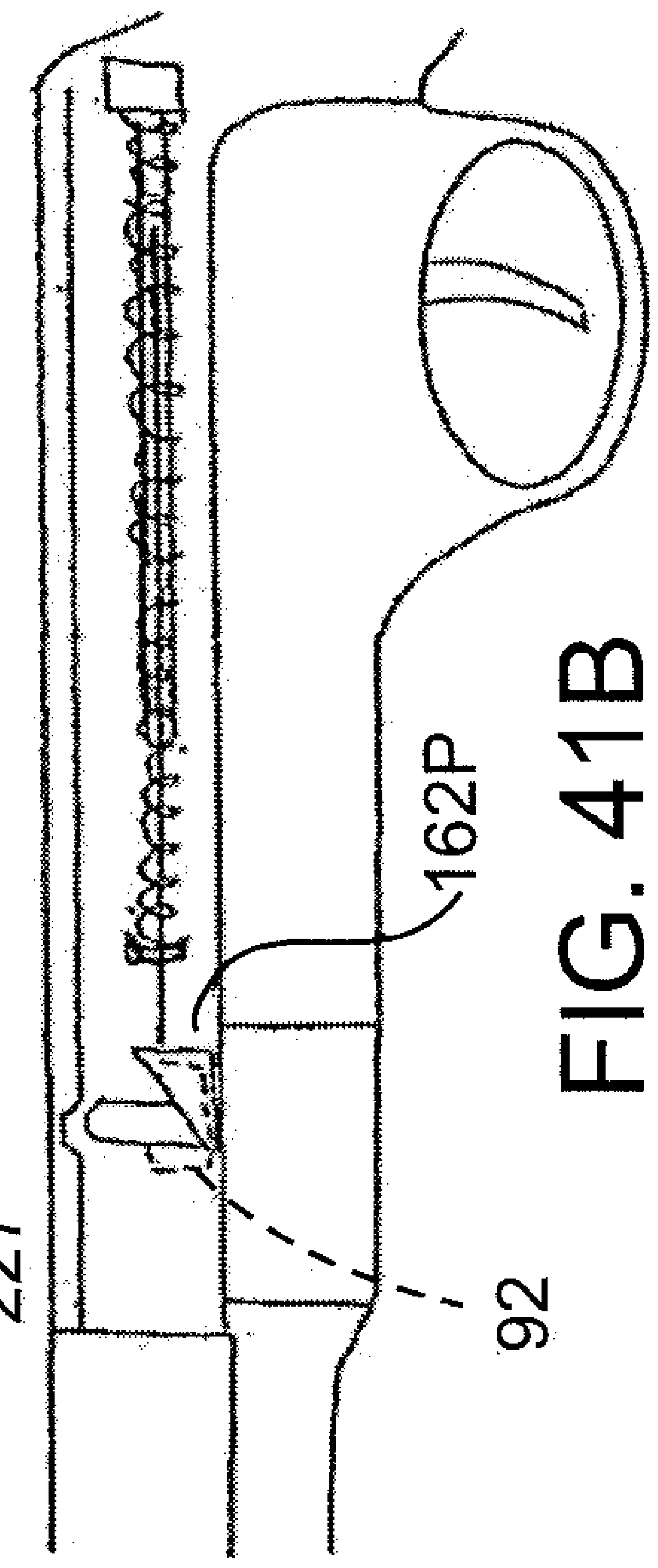


FIG. 41B

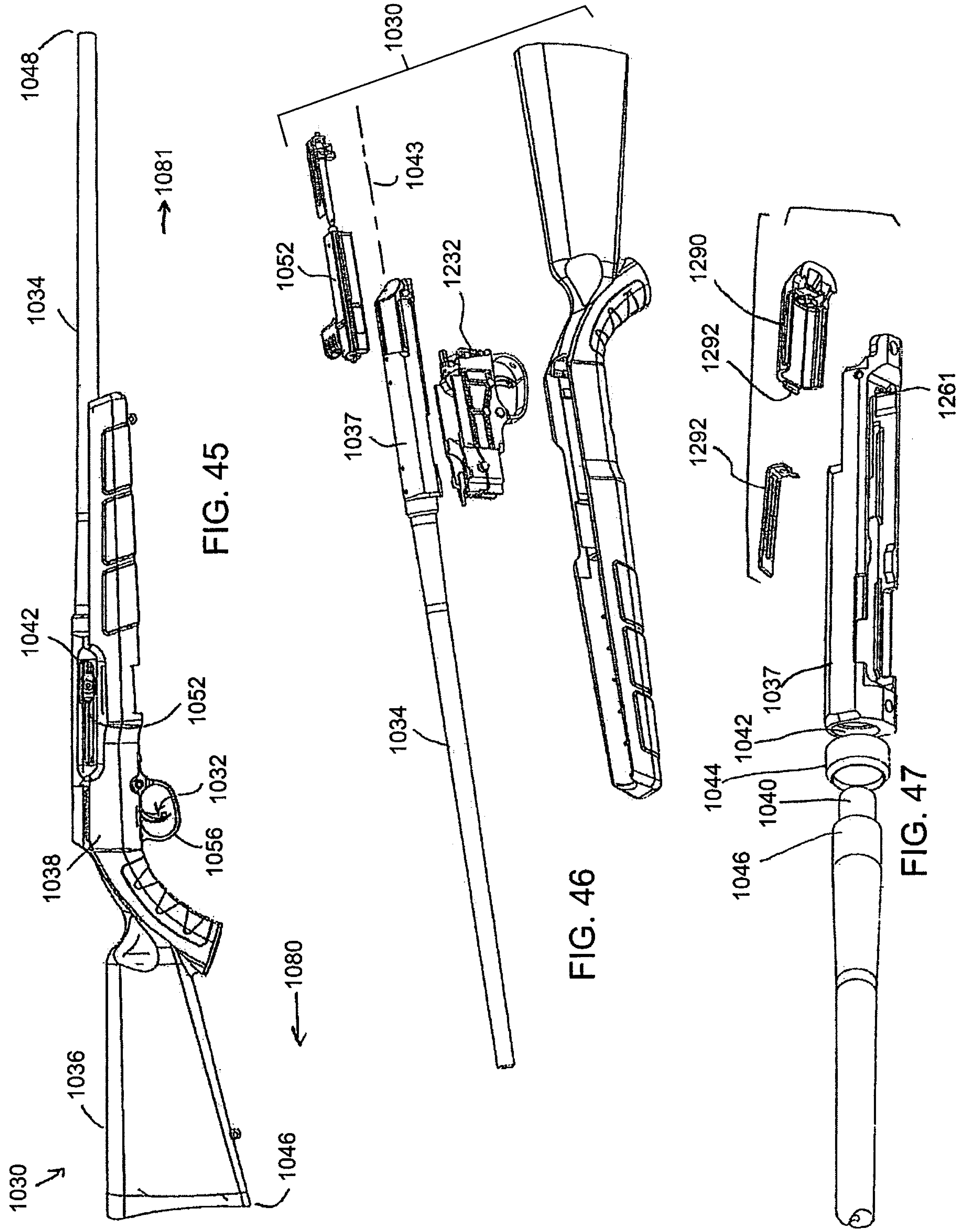


FIG. 45

FIG. 46

FIG. 47

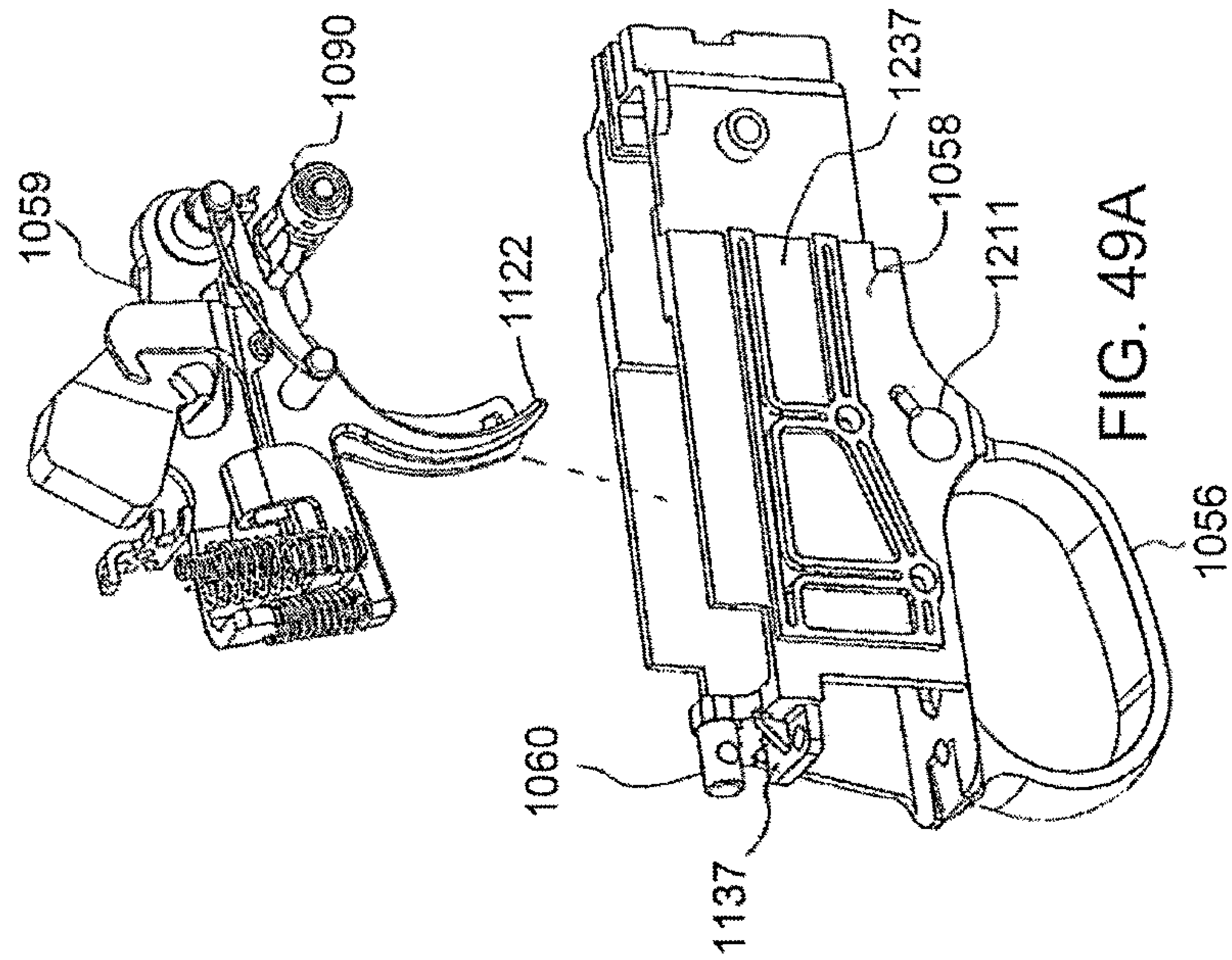


FIG. 49A

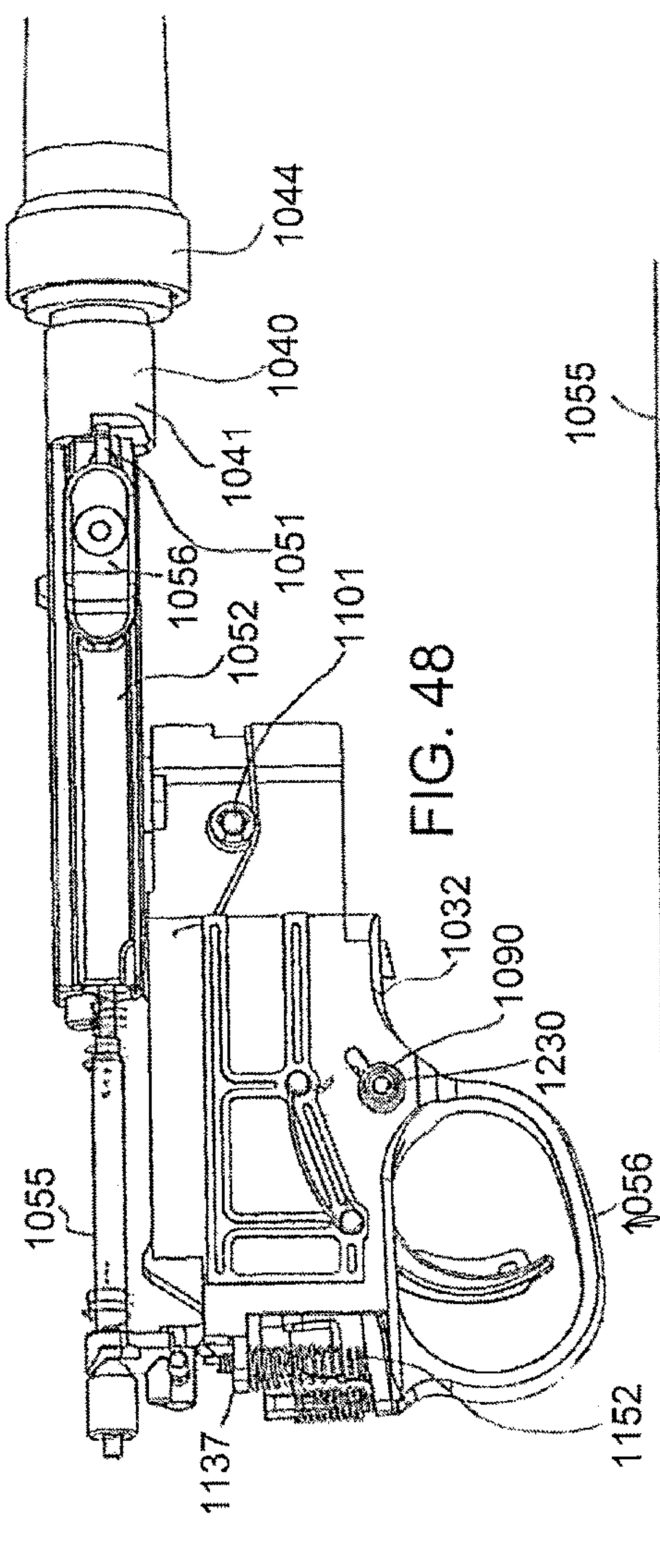


FIG. 48

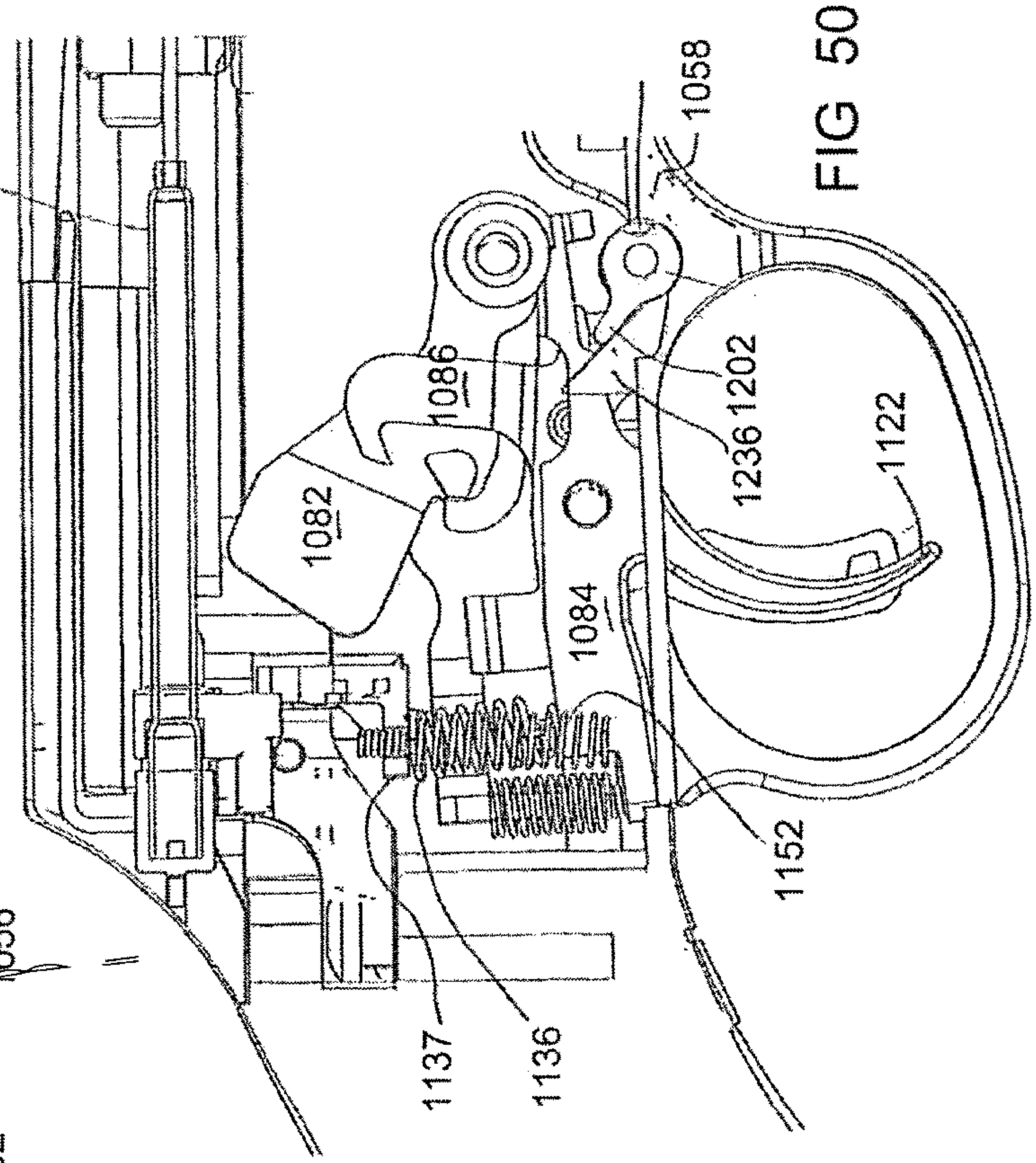


FIG 50

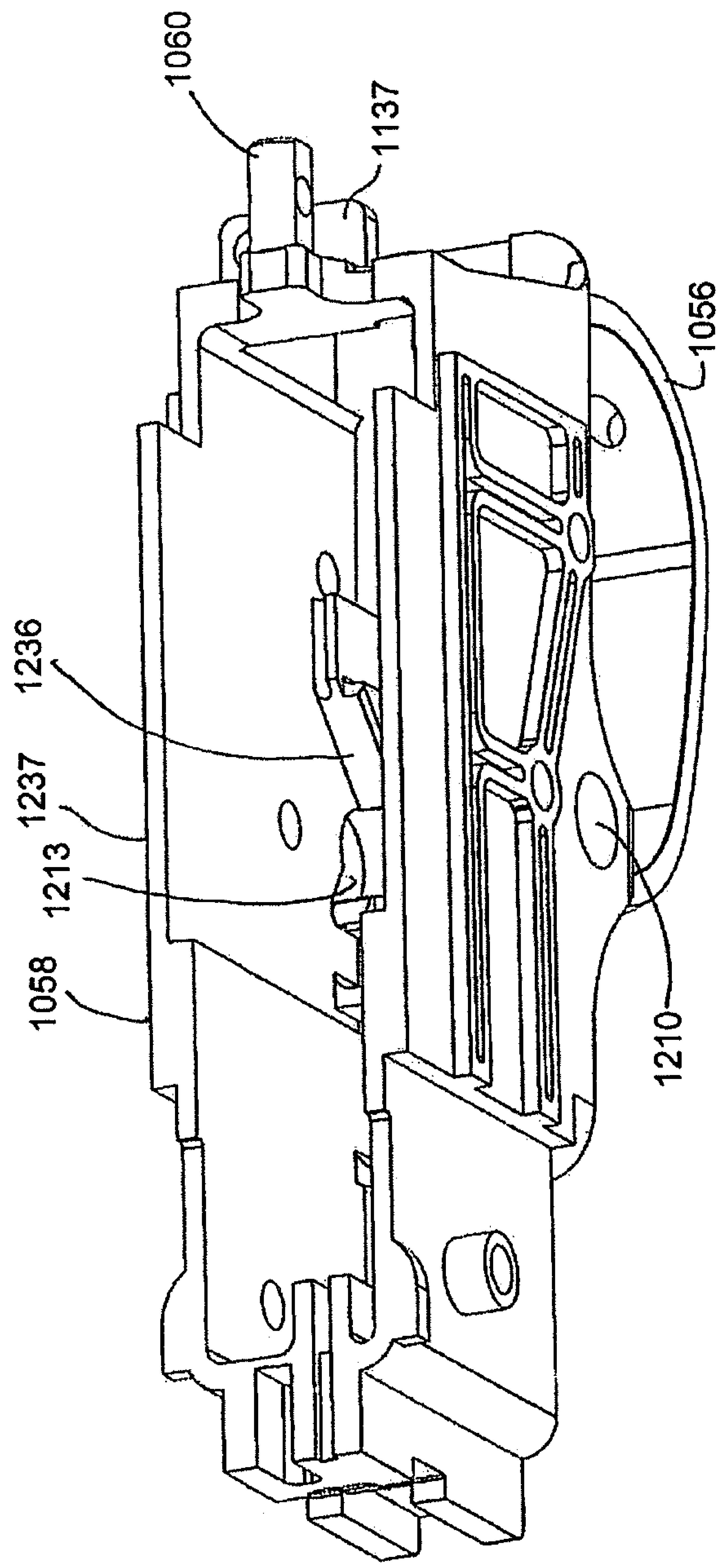


FIG. 49B

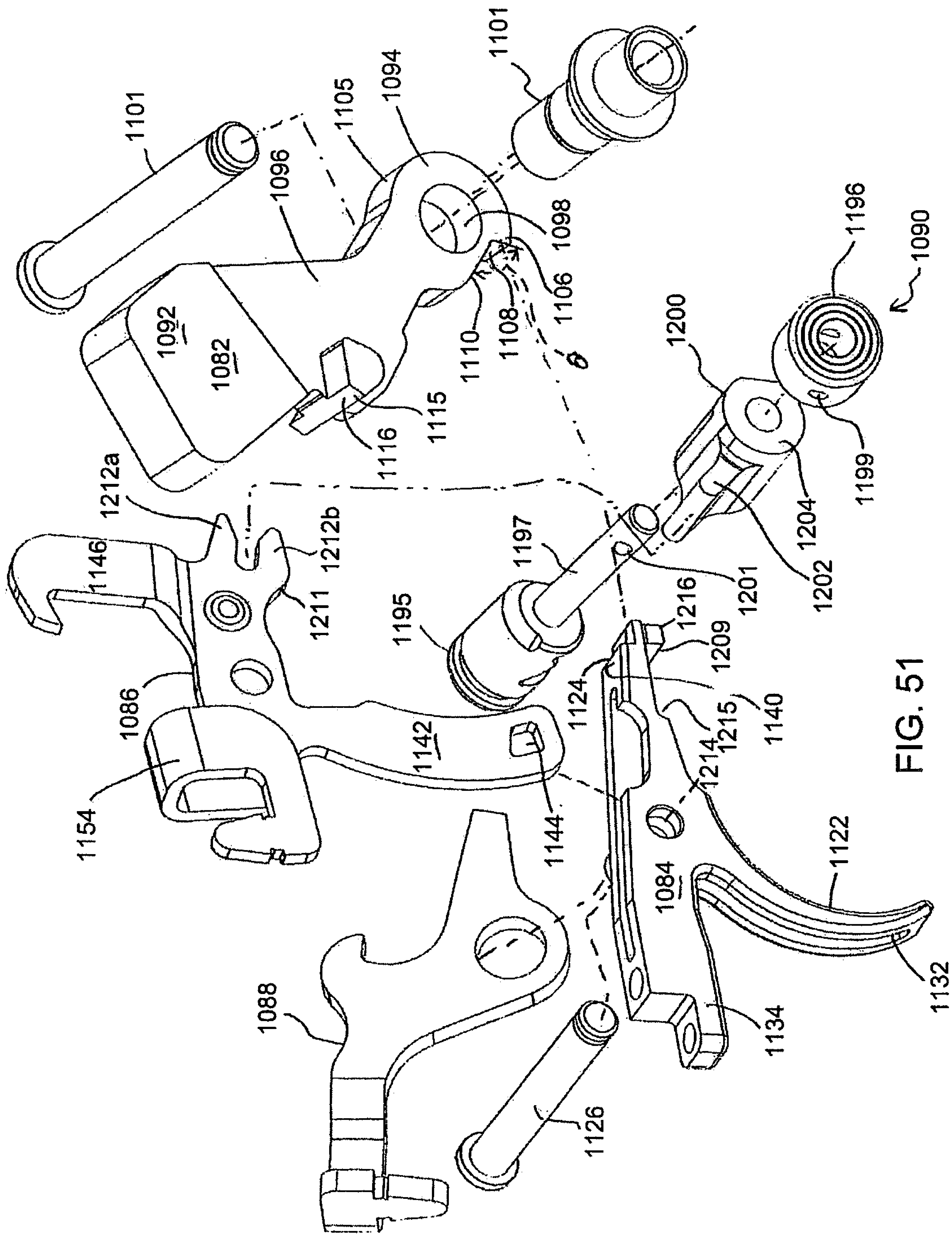
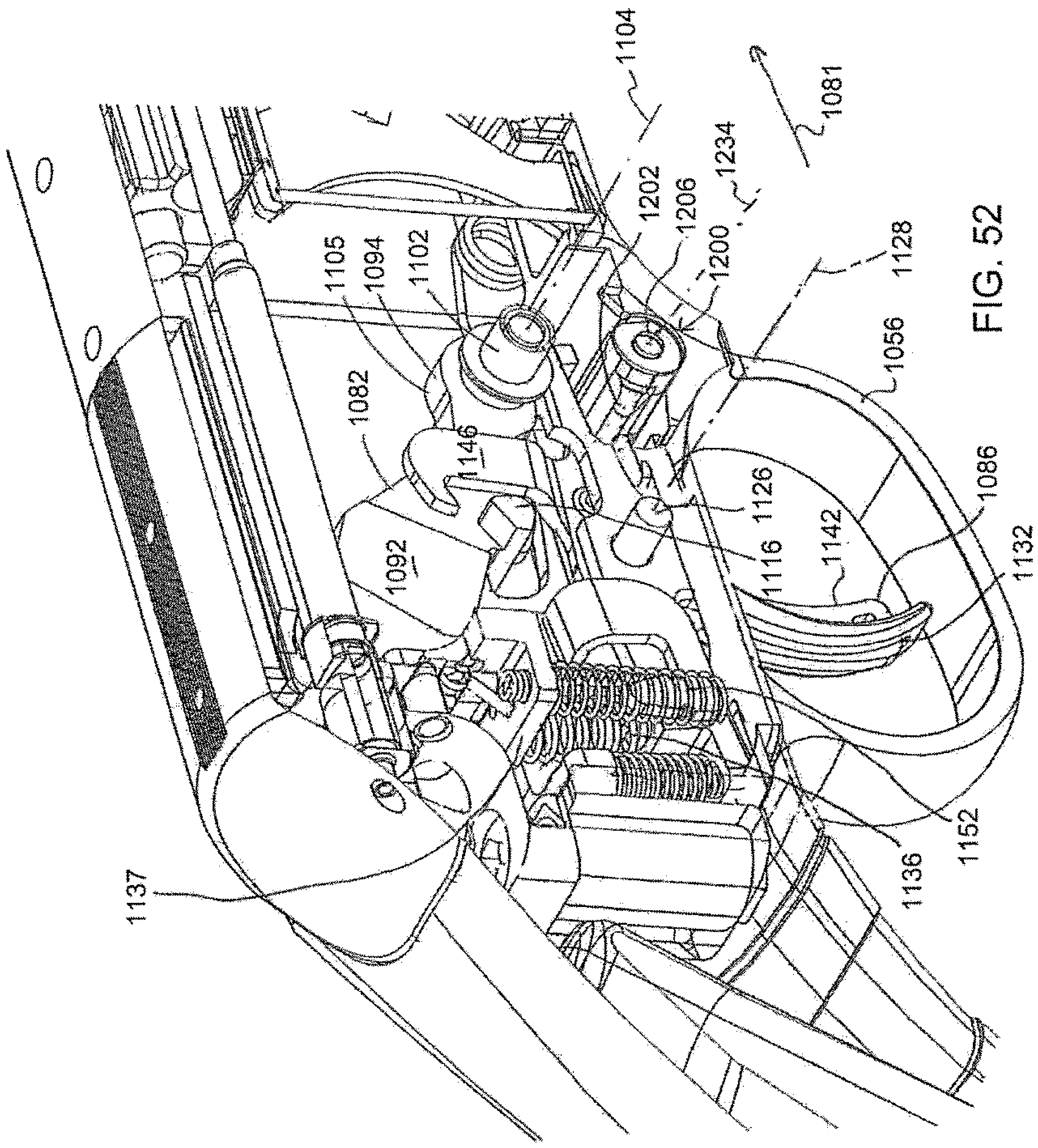

1101
1105
1094
1101
1096
1098
1106
1108
1110
1092
1082
1115
1116
θ
1196
1090
1200
1199
1204
1202
1201
1197
1195
1216
1209
1124
1215
1140
1214
1212a
1212b
1146
1211
1086
1154
1142
1144
1122
1084
1132
1134
1088
1126

FIG. 51


1104
1081
1234
1202
1206
1200
1128
1105
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1082
1146
1126
1086
1092
1116
1142
1132
1137
1136
1152

FIG. 52

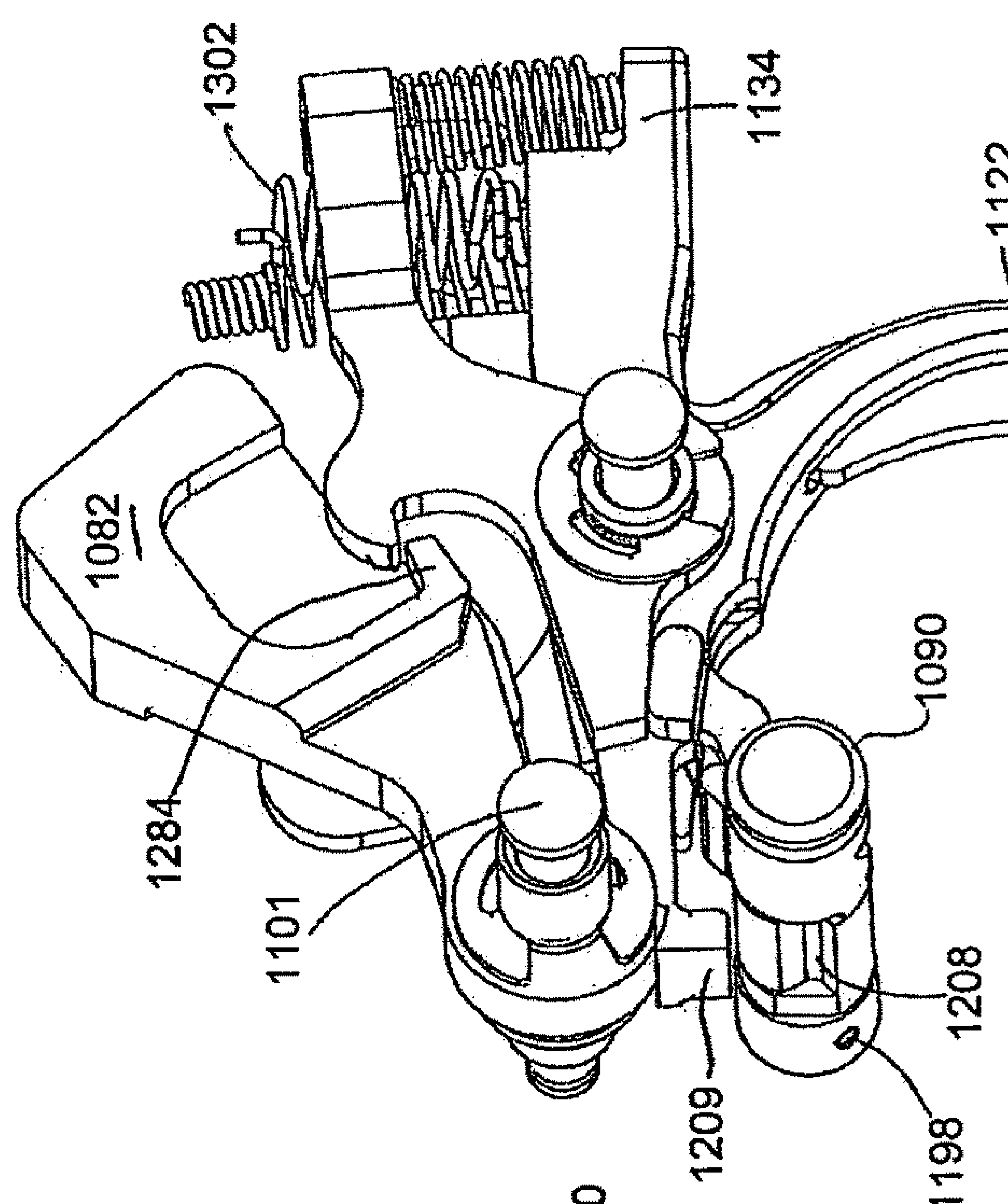


FIG. 54

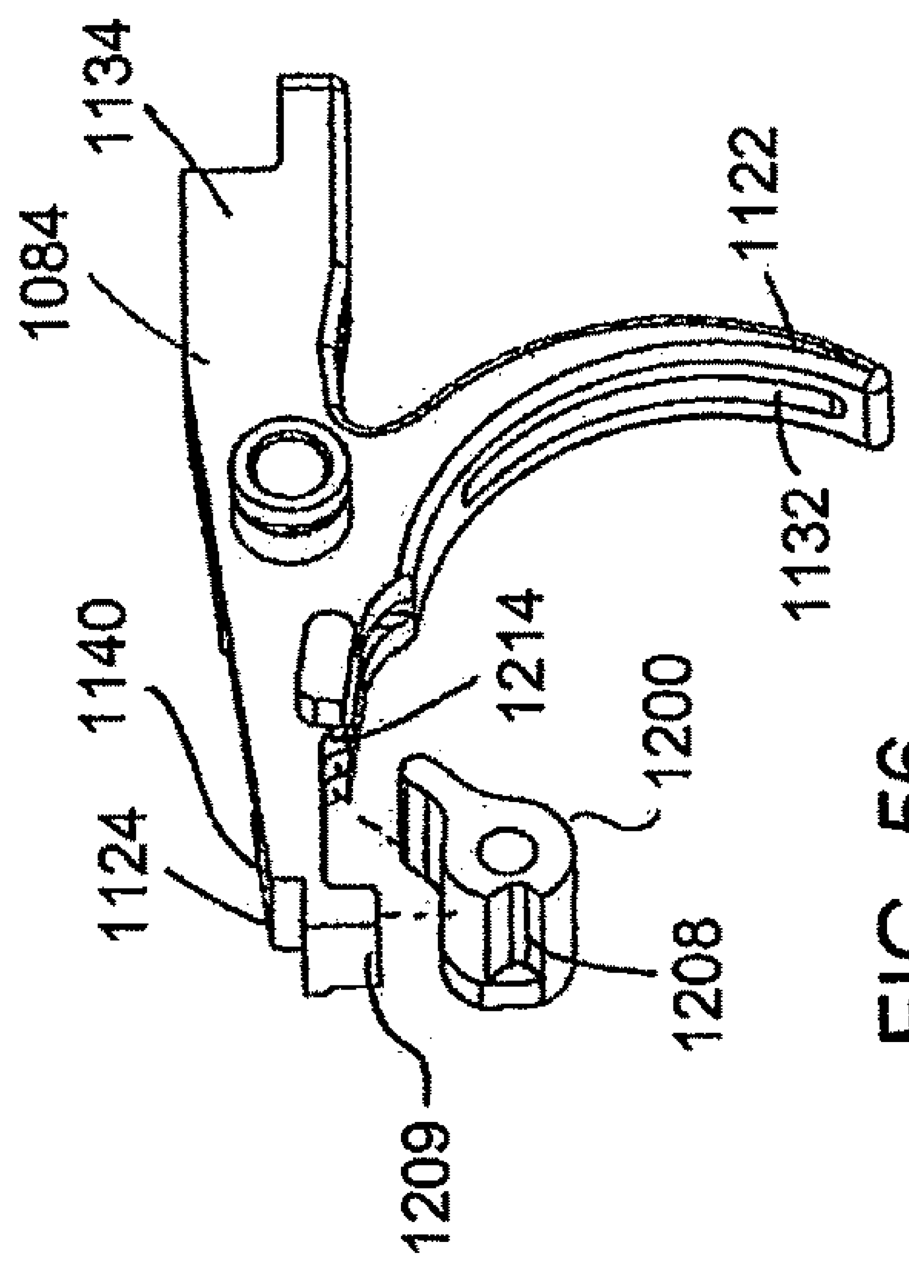


FIG. 56

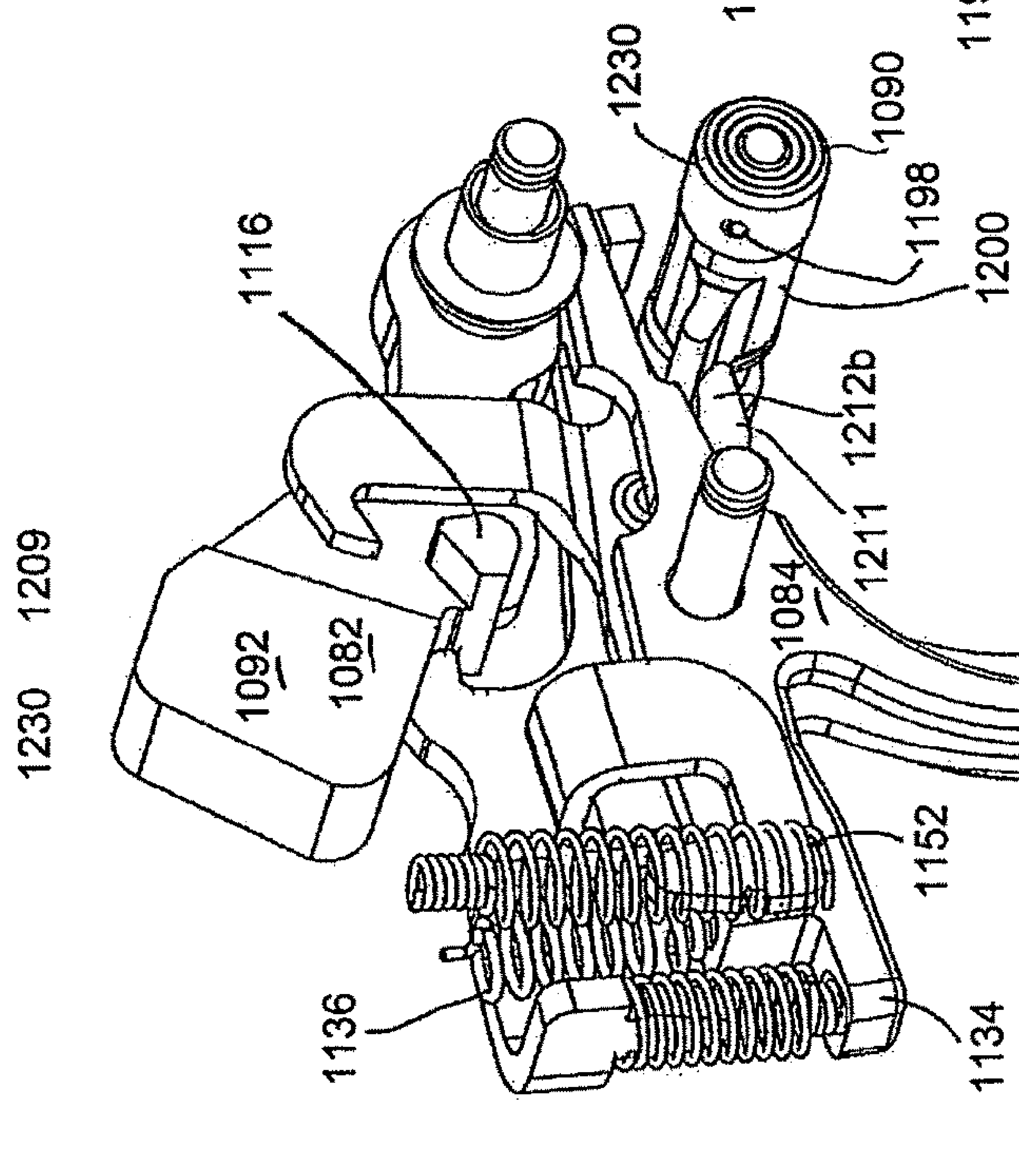


FIG. 53

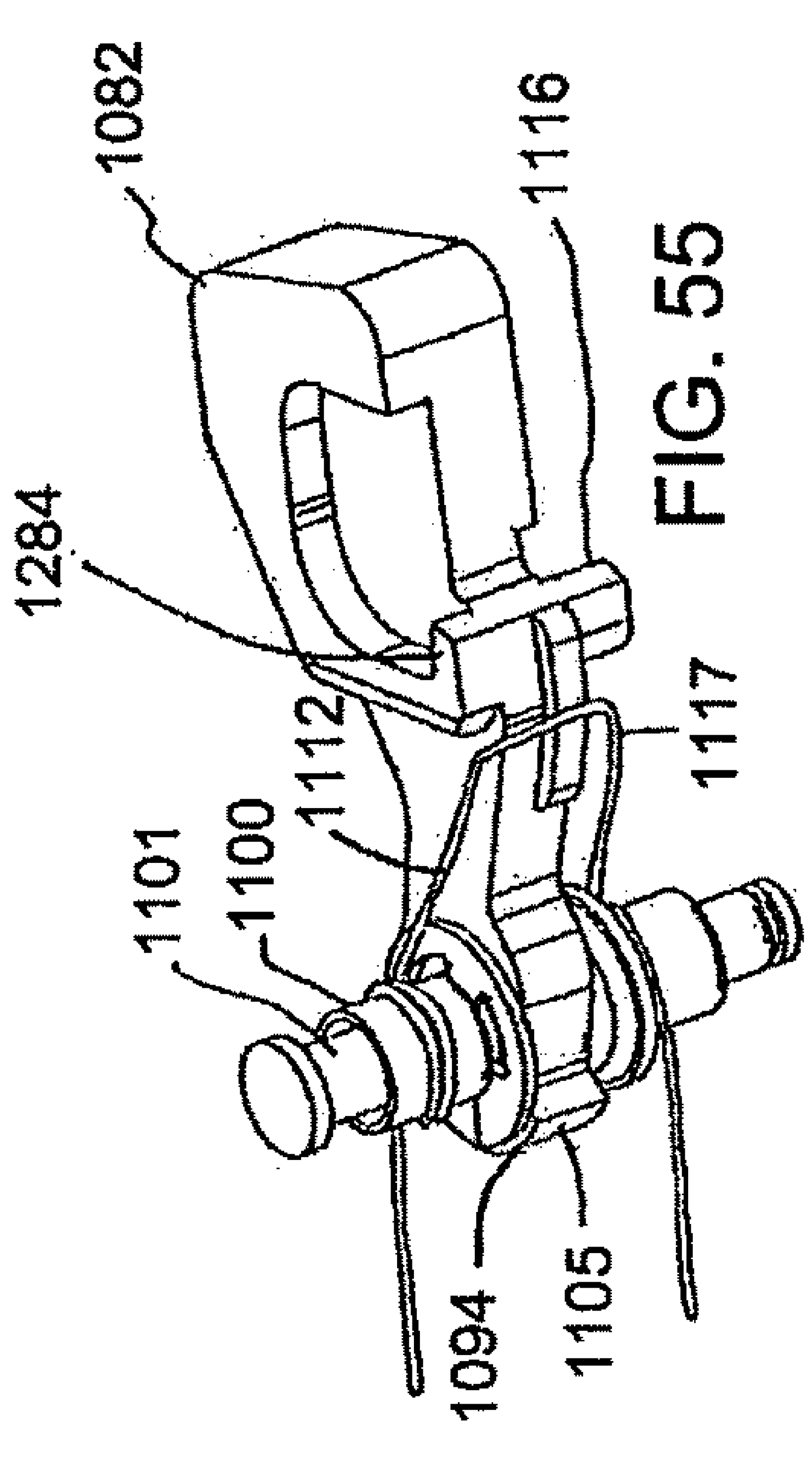
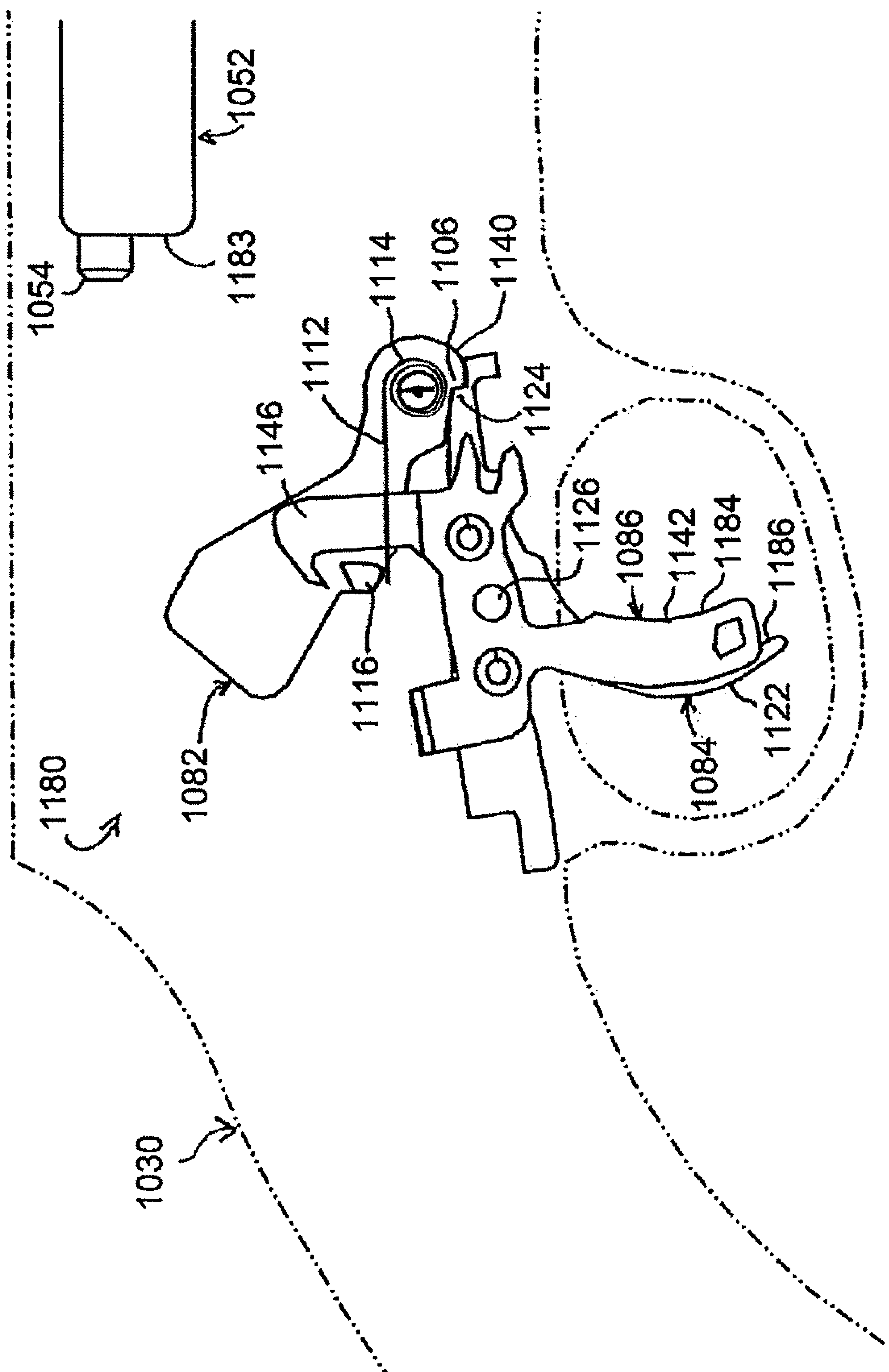


FIG. 55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1054
1183
1114
1106
1140
1112
1124
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1126
1086
1142
1184
1186
1116
1122
1084
1082
1180
1030

FIG. 57

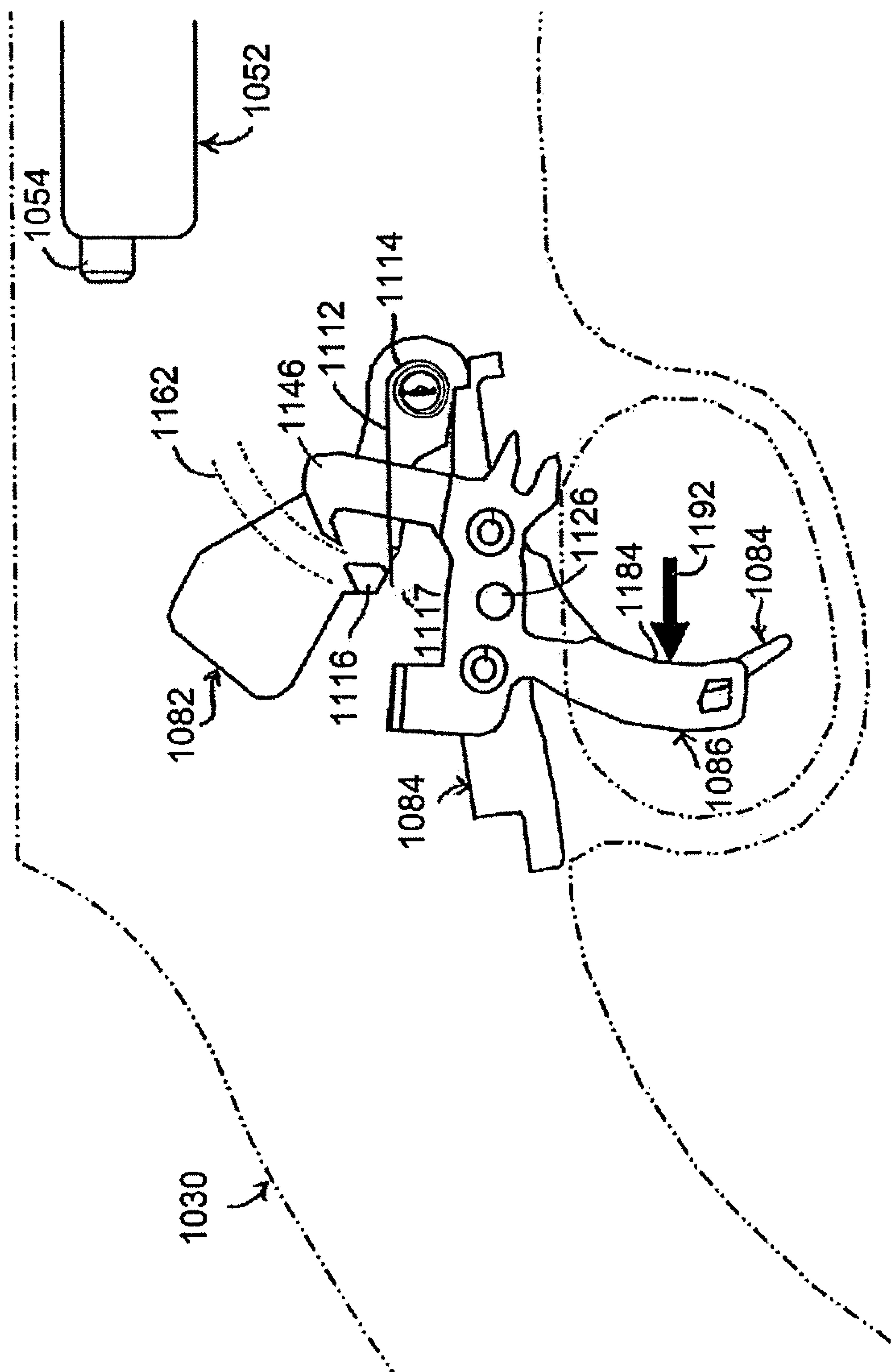

1052
1054
1114
1112
1146
1162
1126
1192
1184
1084
1117
1116
1082
1086
1084
1030

FIG. 58

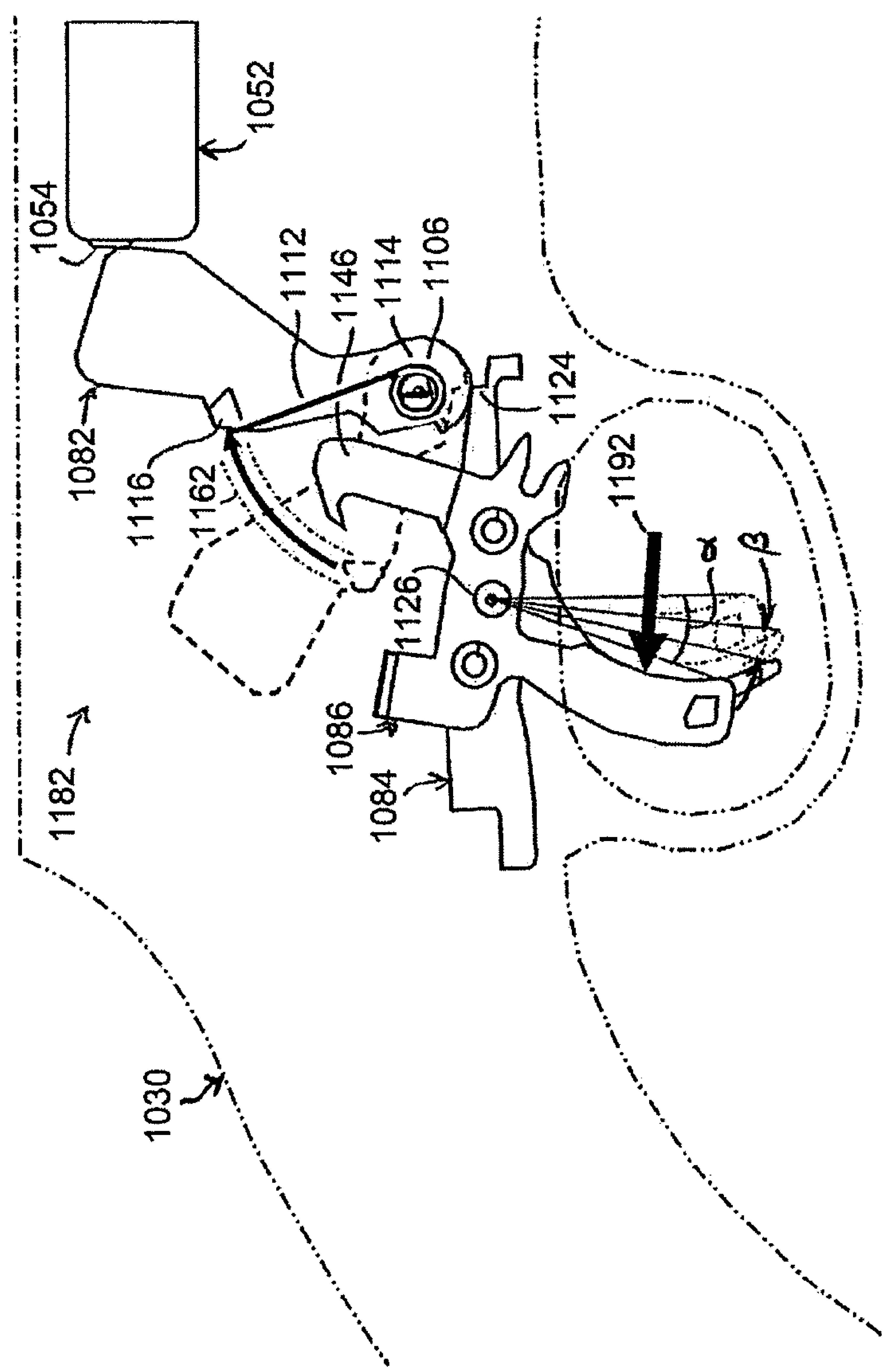
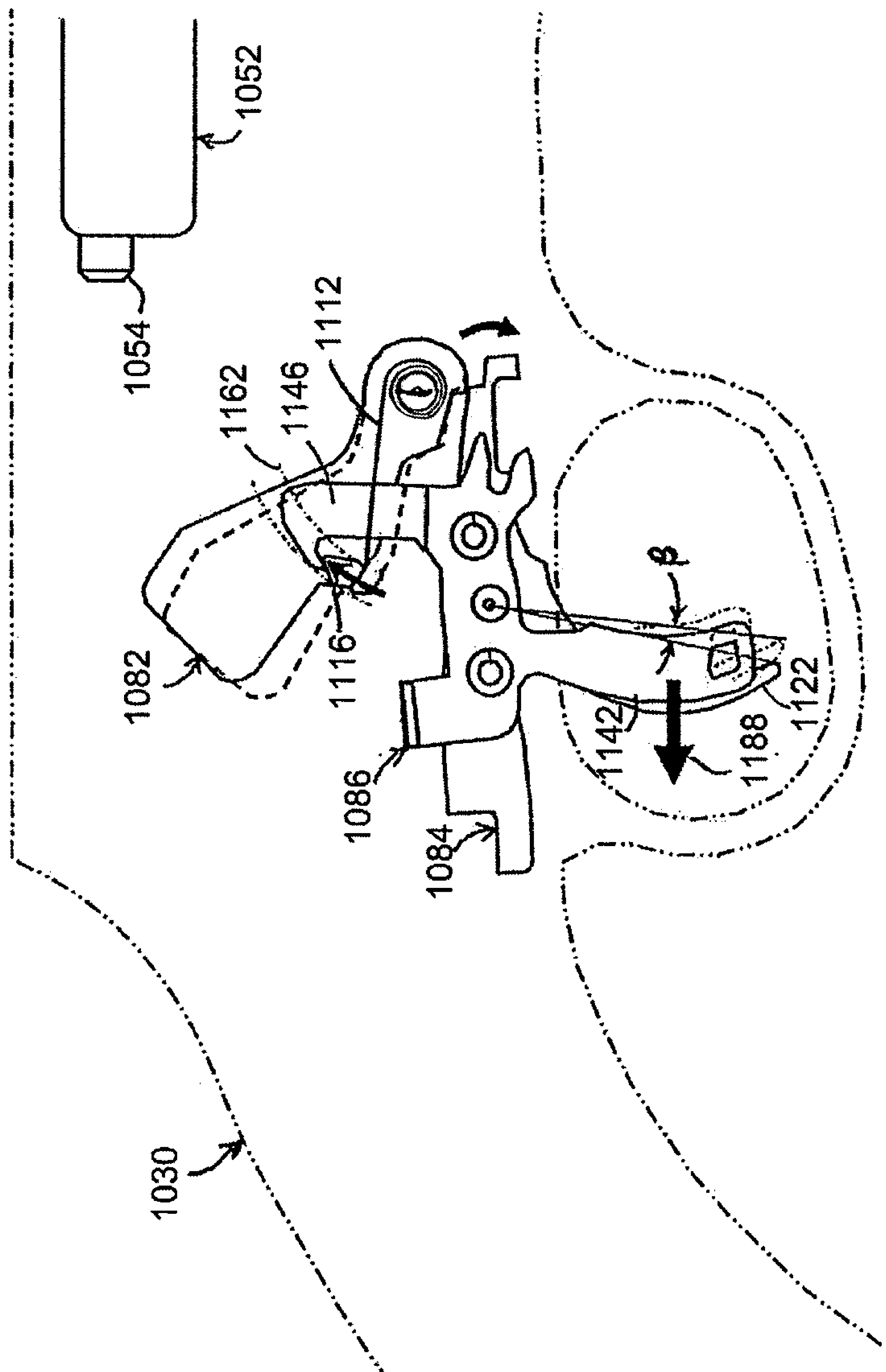

1052
1054
1112
1146
1114
1106
1124
1082
1116
1162
1192
α
β
1126
1086
1084
1182
1030

FIG. 59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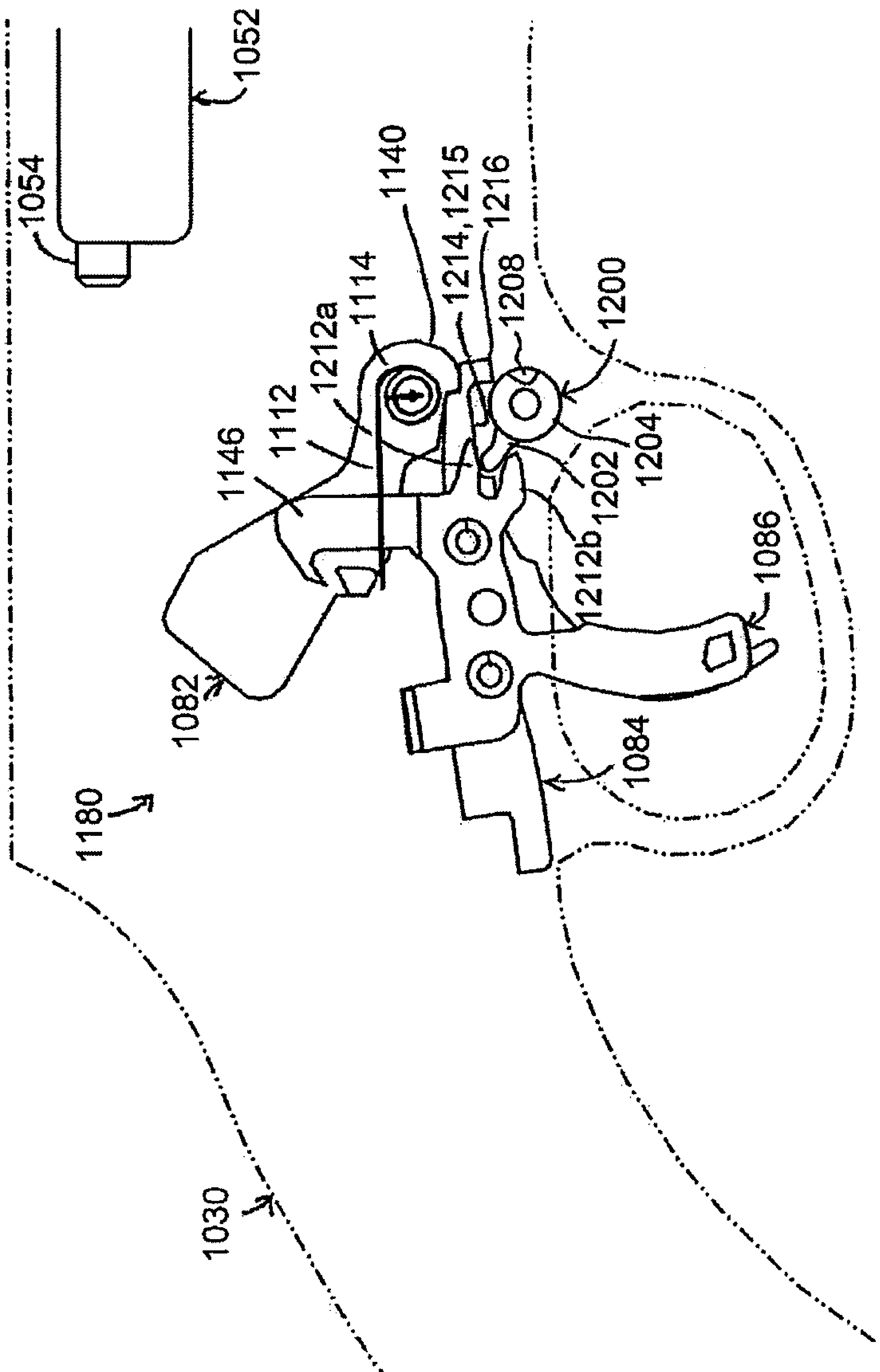
1162
1146
1082
1116
1086
1084
β
1142
1188
1122
1030

FIG. 60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1054
1140
1214,1215
1216
1208
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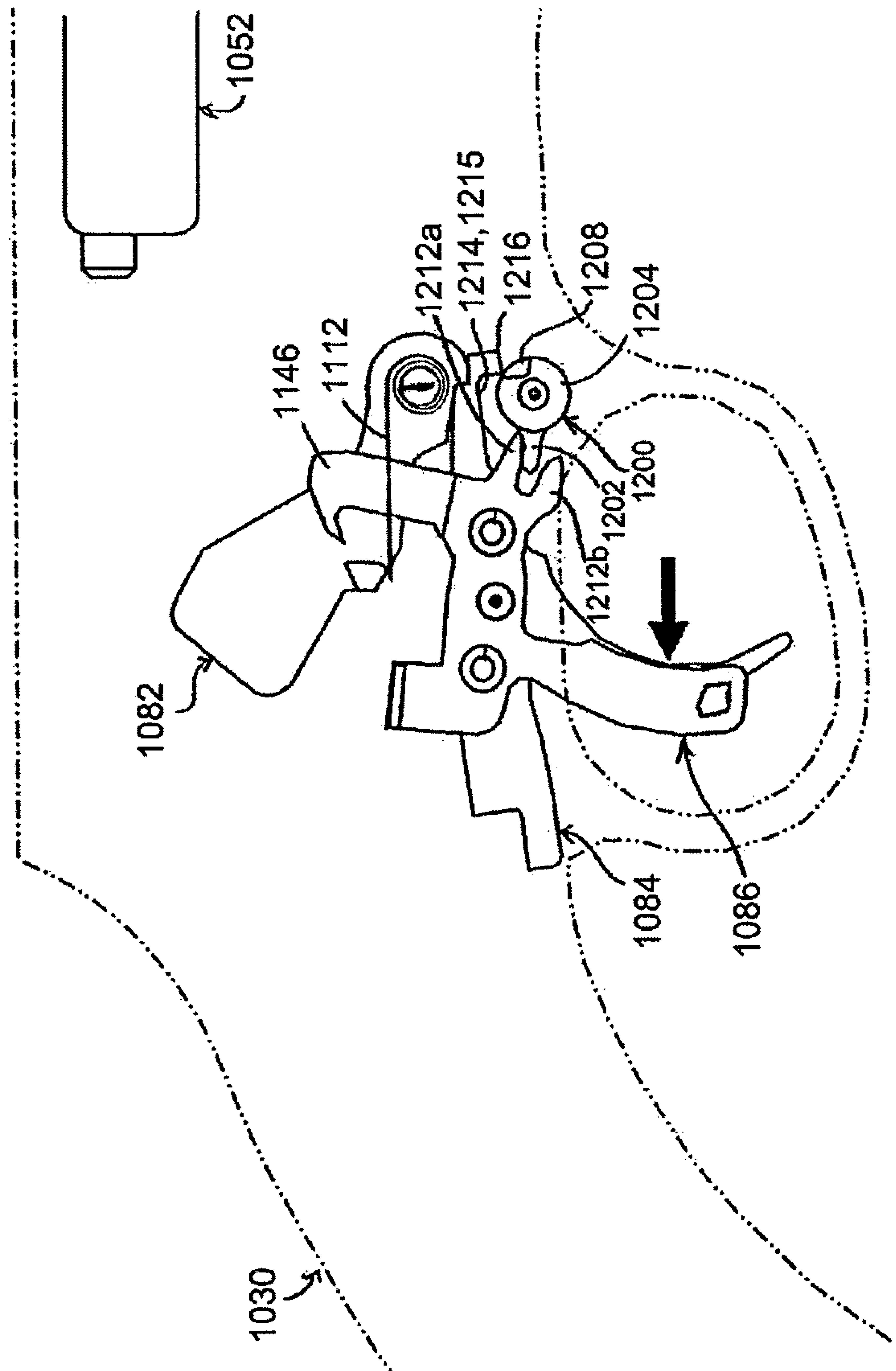
1114
1212a
1112
1146
1204
1202
1212b
1086
1082
1084
1180
1030

FIG. 61


1052
1212a
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1208
1204
1146
1112
1200
1202
1212b
1082
1084
1086
1030

FIG. 62

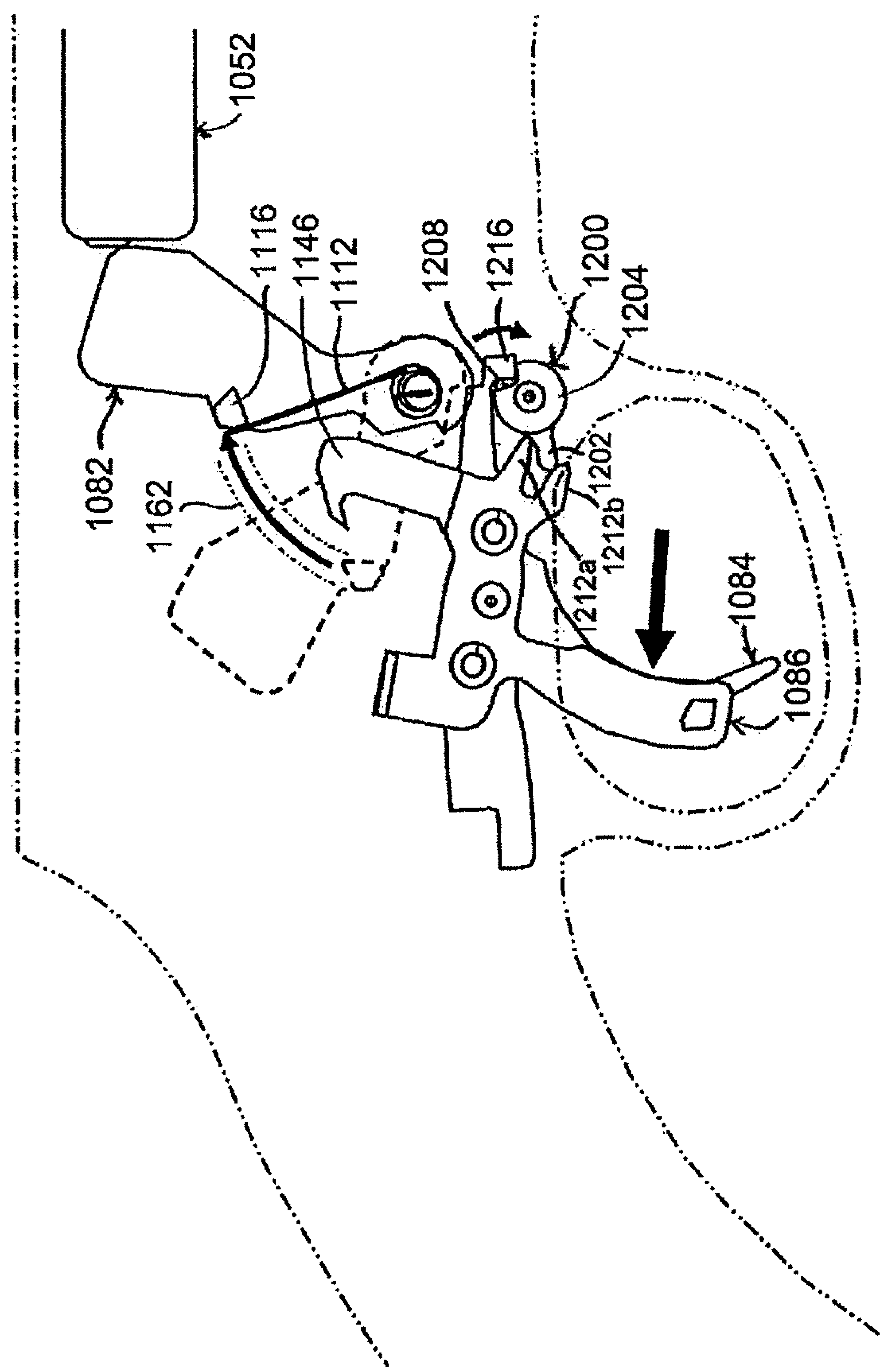
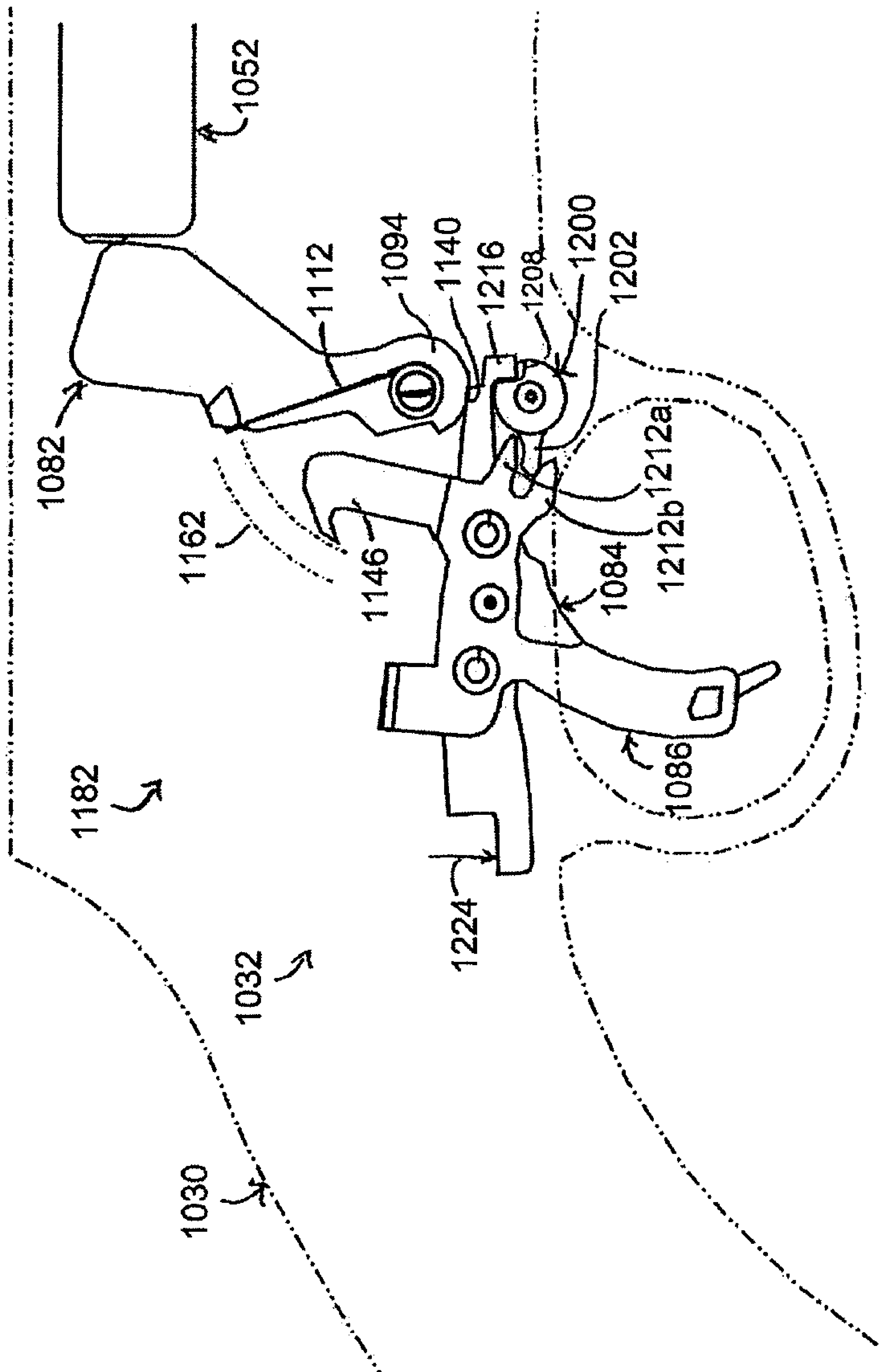

1052
1116
1146
1112
1208
1216
1200
1204
1082
1162
1202
1212b
1212a
1084
1086

FIG. 63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1140
1216
1208
1200
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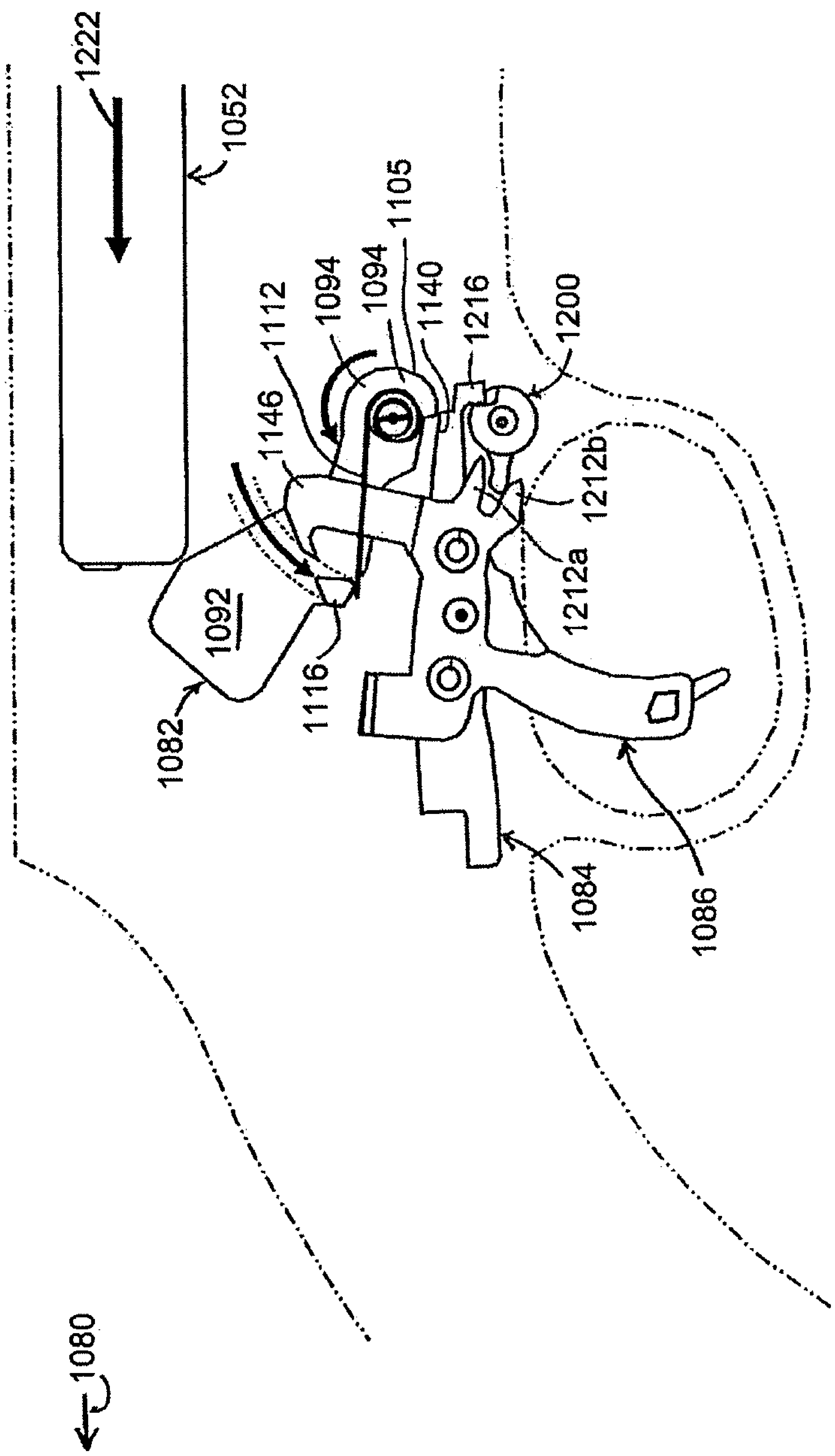
1082
1212a
1212b
1162
1146
1084
1086
1182
1224
1032
1030

FIG. 64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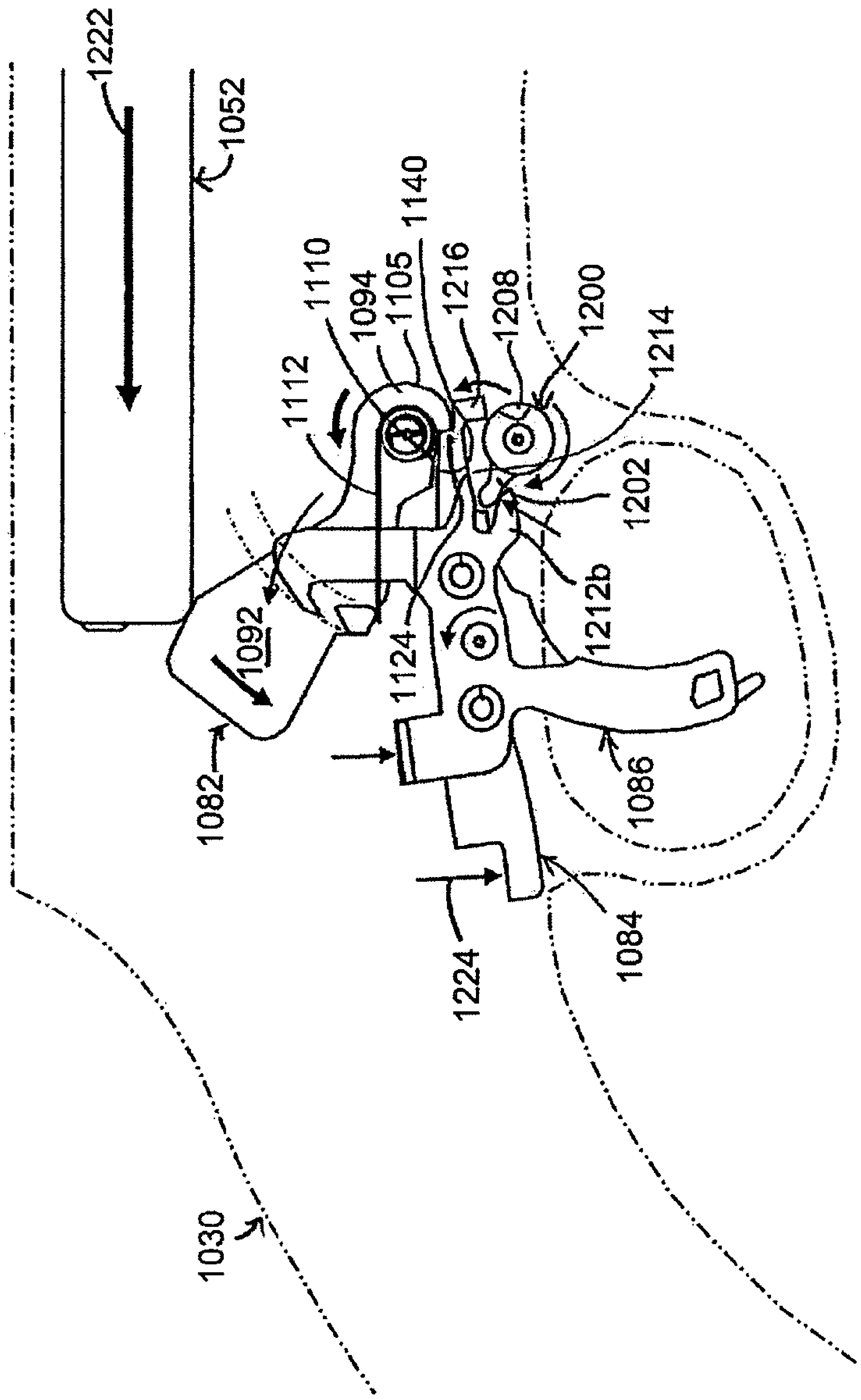
1105
1094
1094
1112
1140
1216
1200
1146
1212b
1212a
1092
1116
1082
1084
1086
1080

FIG. 65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1105
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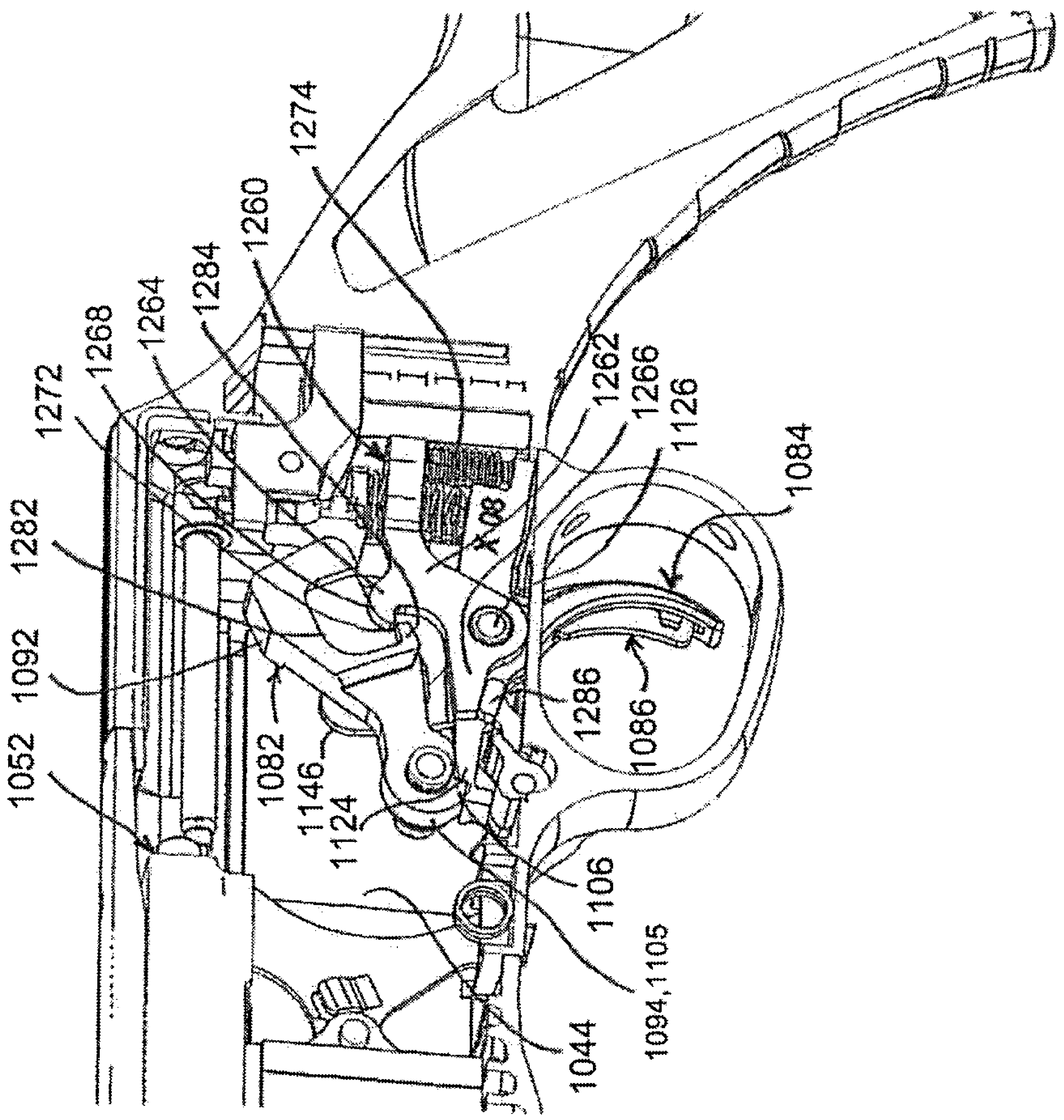
1124
1082
1086
1084
1224
1030

FIG. 66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1268
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1282
1092
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1082
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1106
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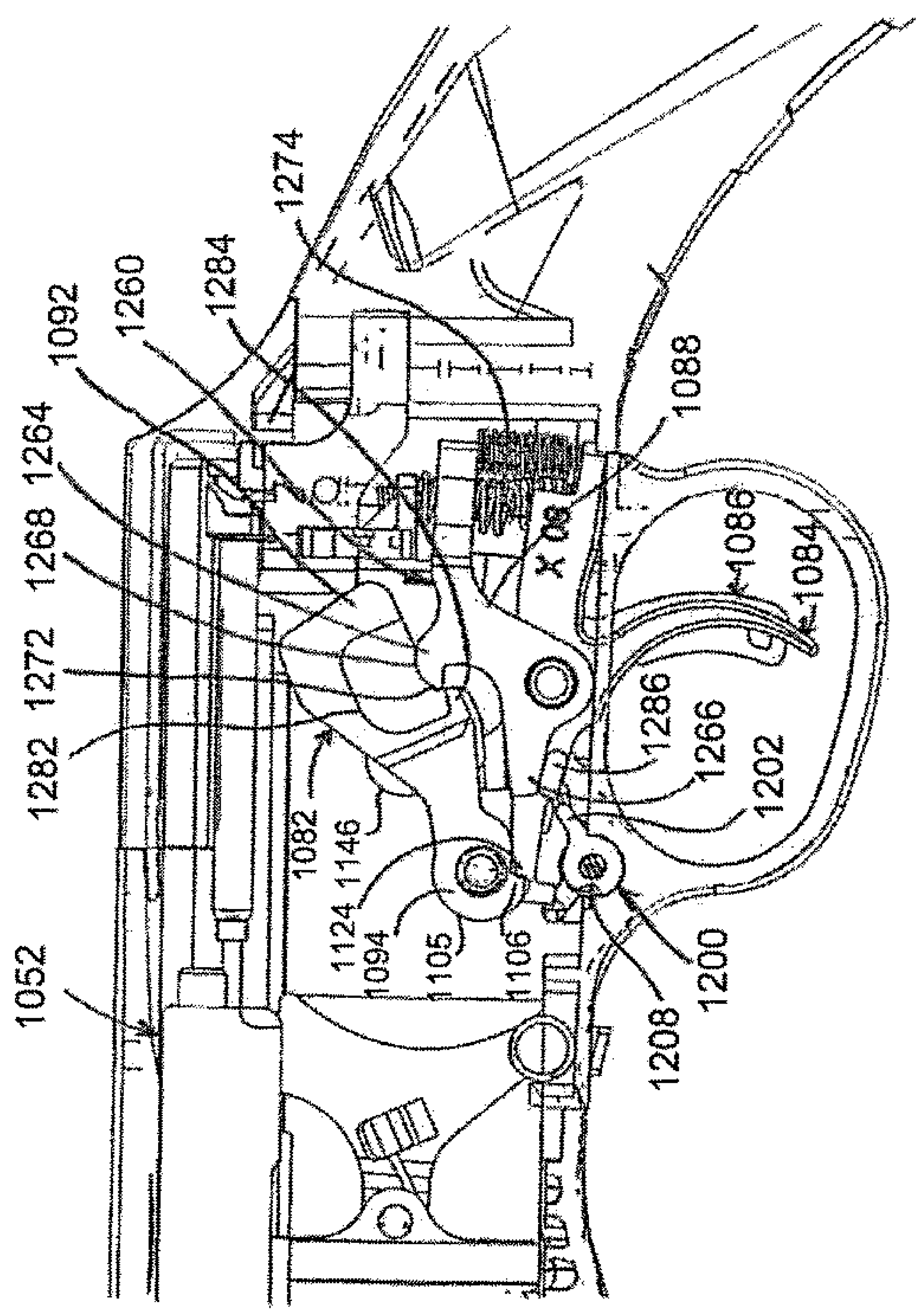
1044
1262
1266
1126
1084
1286
1086

FIG. 67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1282
1088
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1084
1286
1266
1202
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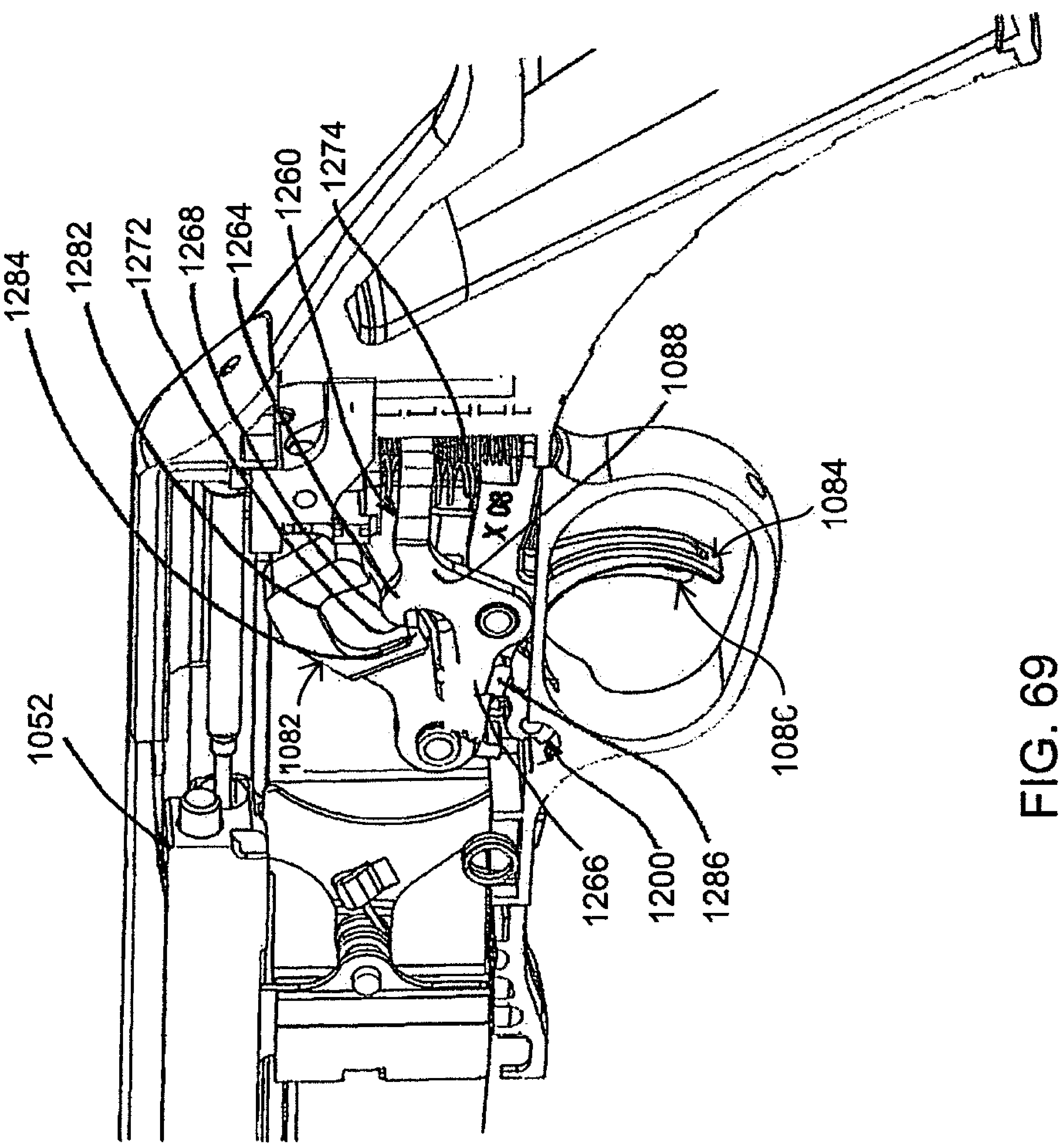
1146
1124
1094
1105
1106
1208
1200
1052

FIG. 68


1284
1282
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1268
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1260
1274
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1084
1086
1286
1200
1266
1082
1052
80 X

FIG. 69

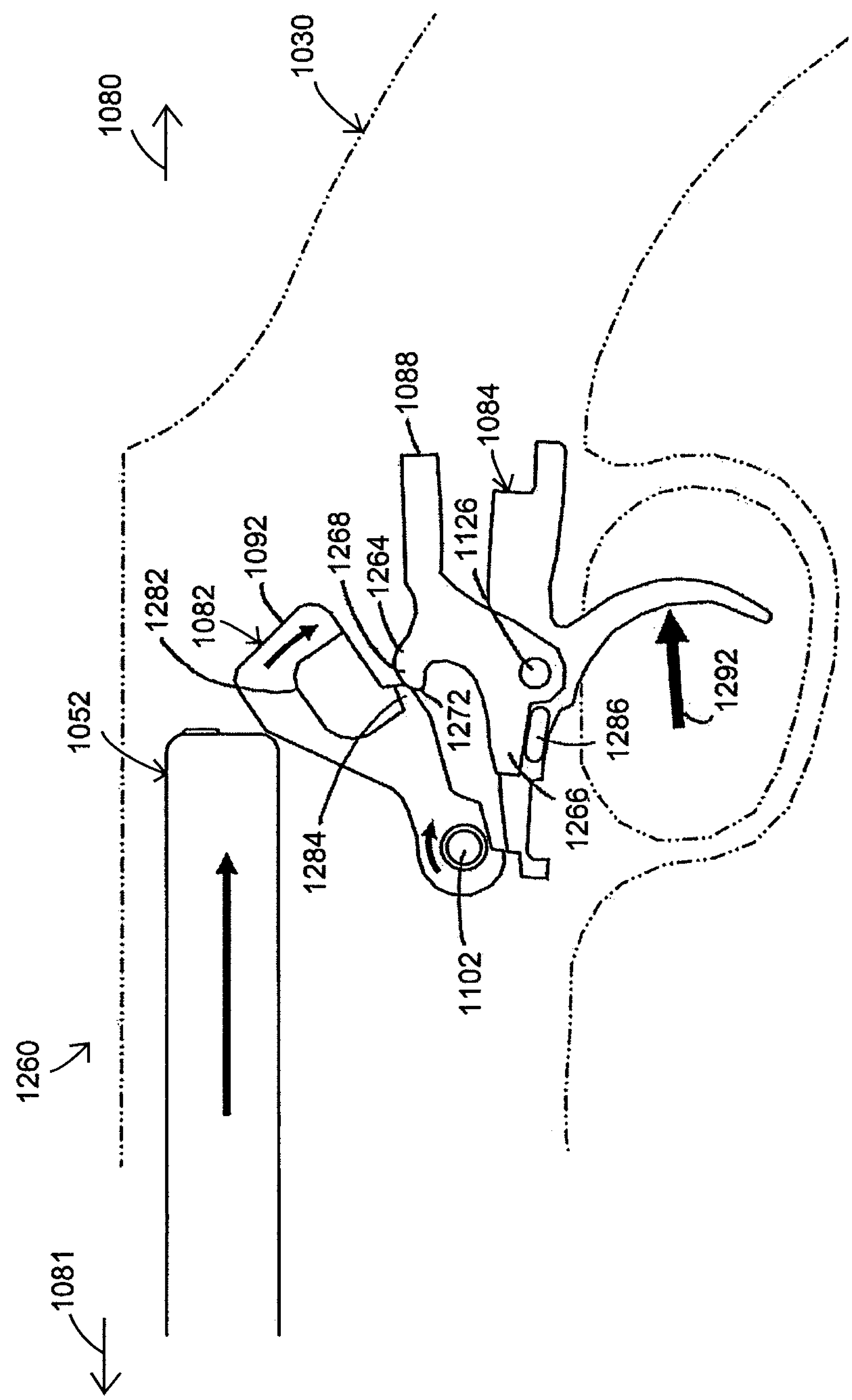

1030
1080
1088
1084
1126
1092
1268
1264
1282
1082
1052
1272
1286
1292
1266
1284
1102
1260
1081

FIG. 70

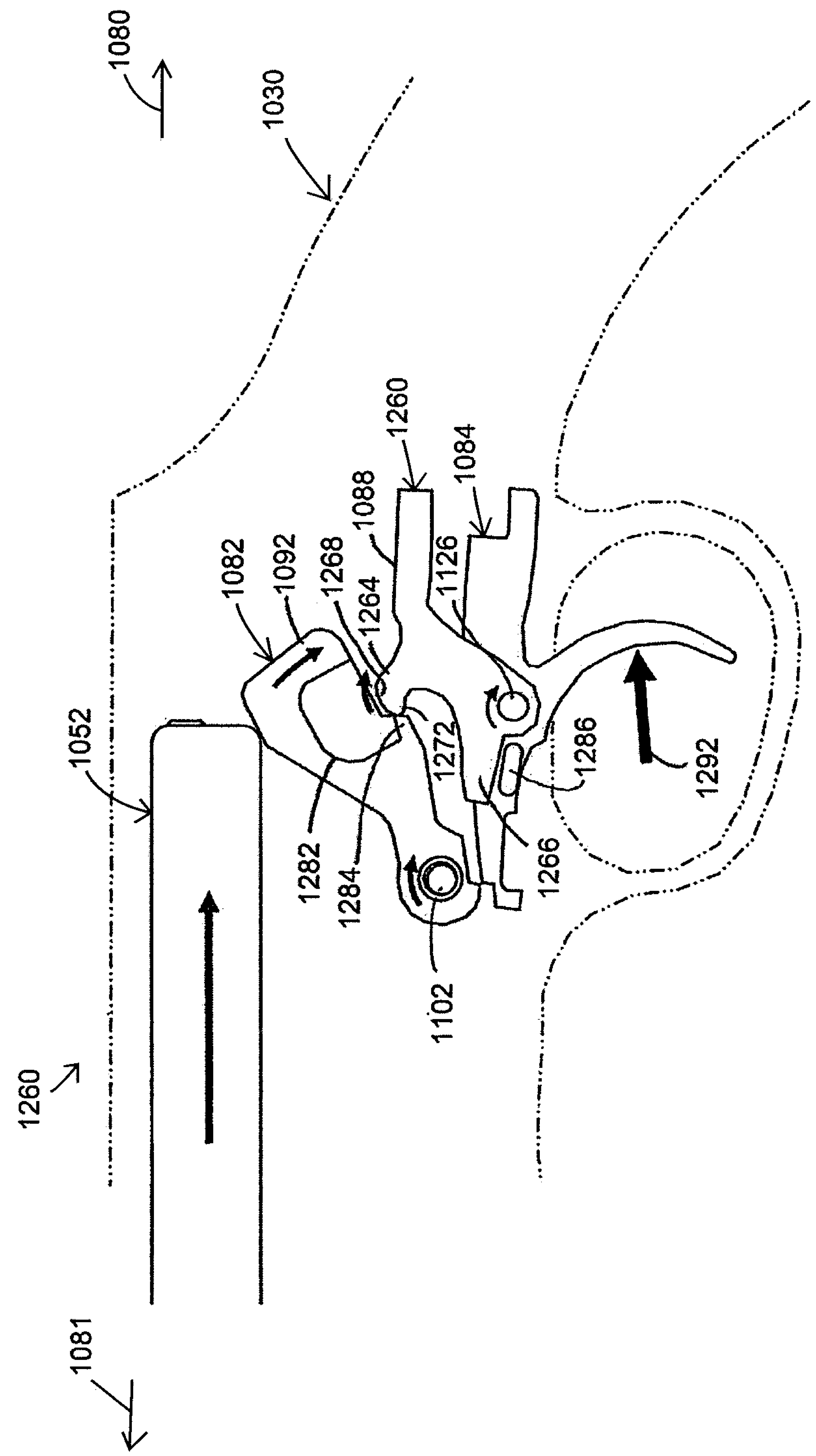

1080
1030
1260
1084
1088
1126
1082
1092
1268
1264
1052
1272
1286
1292
1282
1284
1266
1102
1260
1081

FIG. 71

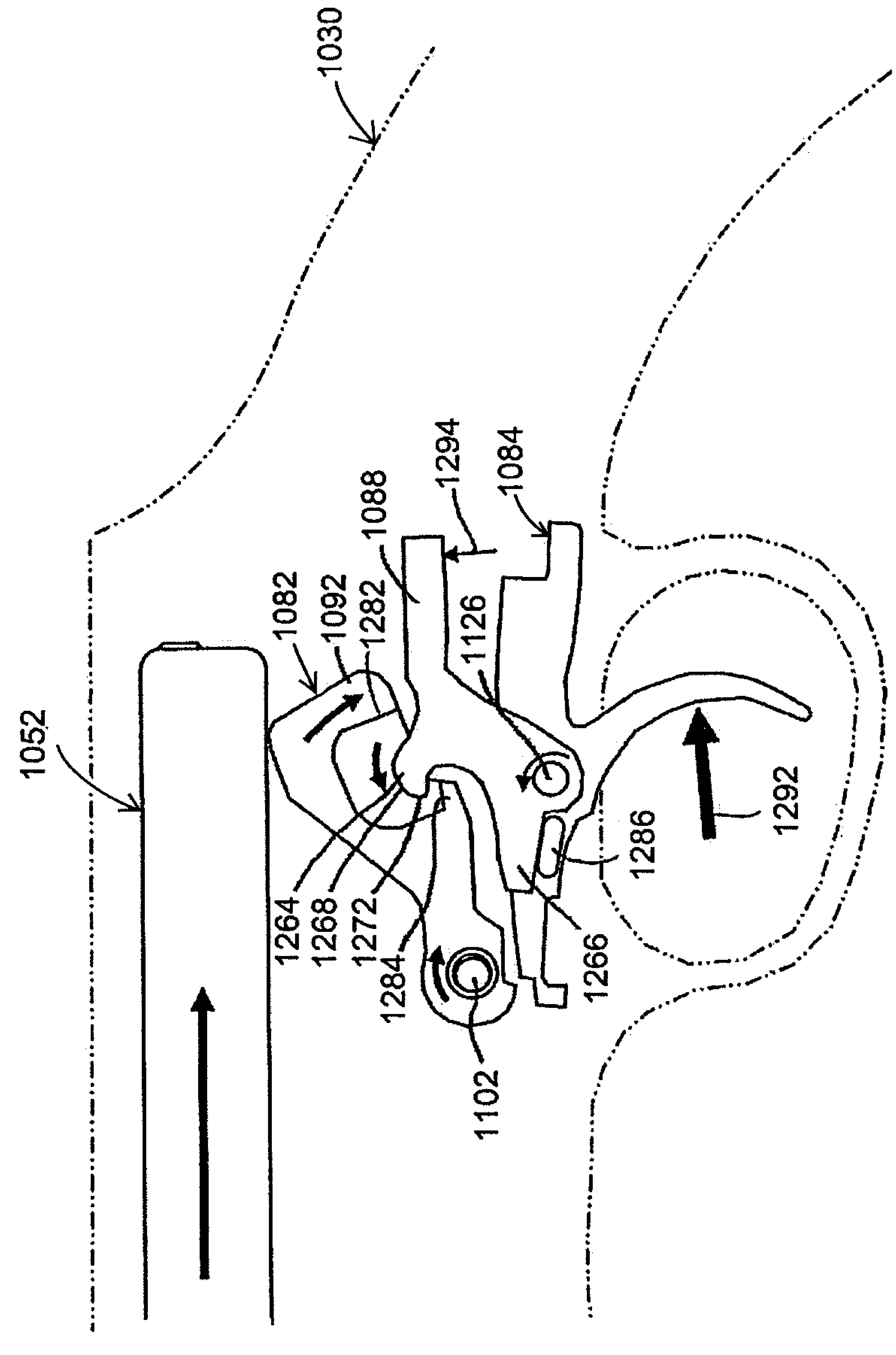

1030
1052
1082
1092
1282
1088
1294
1084
1126
1264
1268
1272
1284
1102
1266
1286
1292

FIG. 72

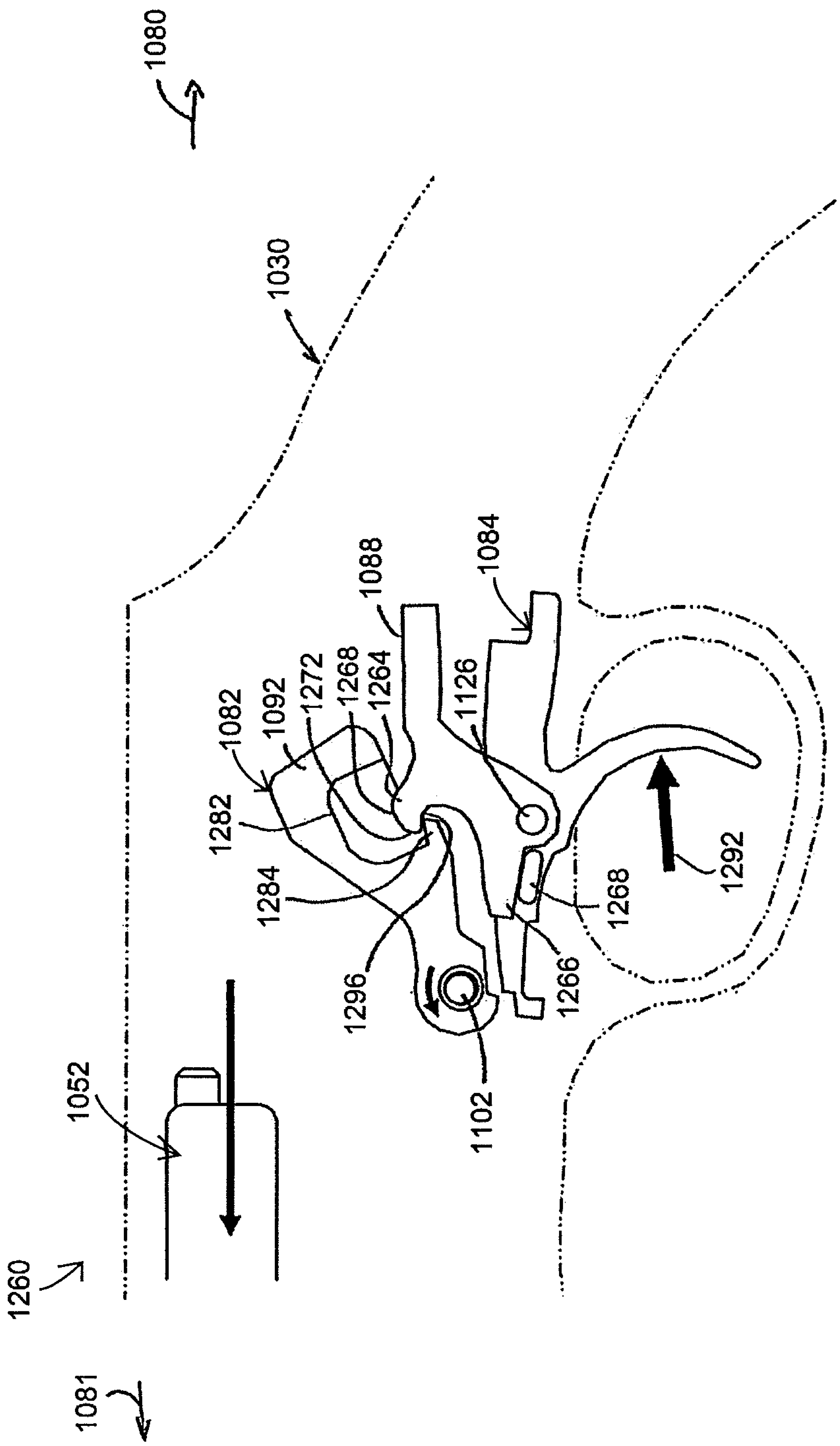
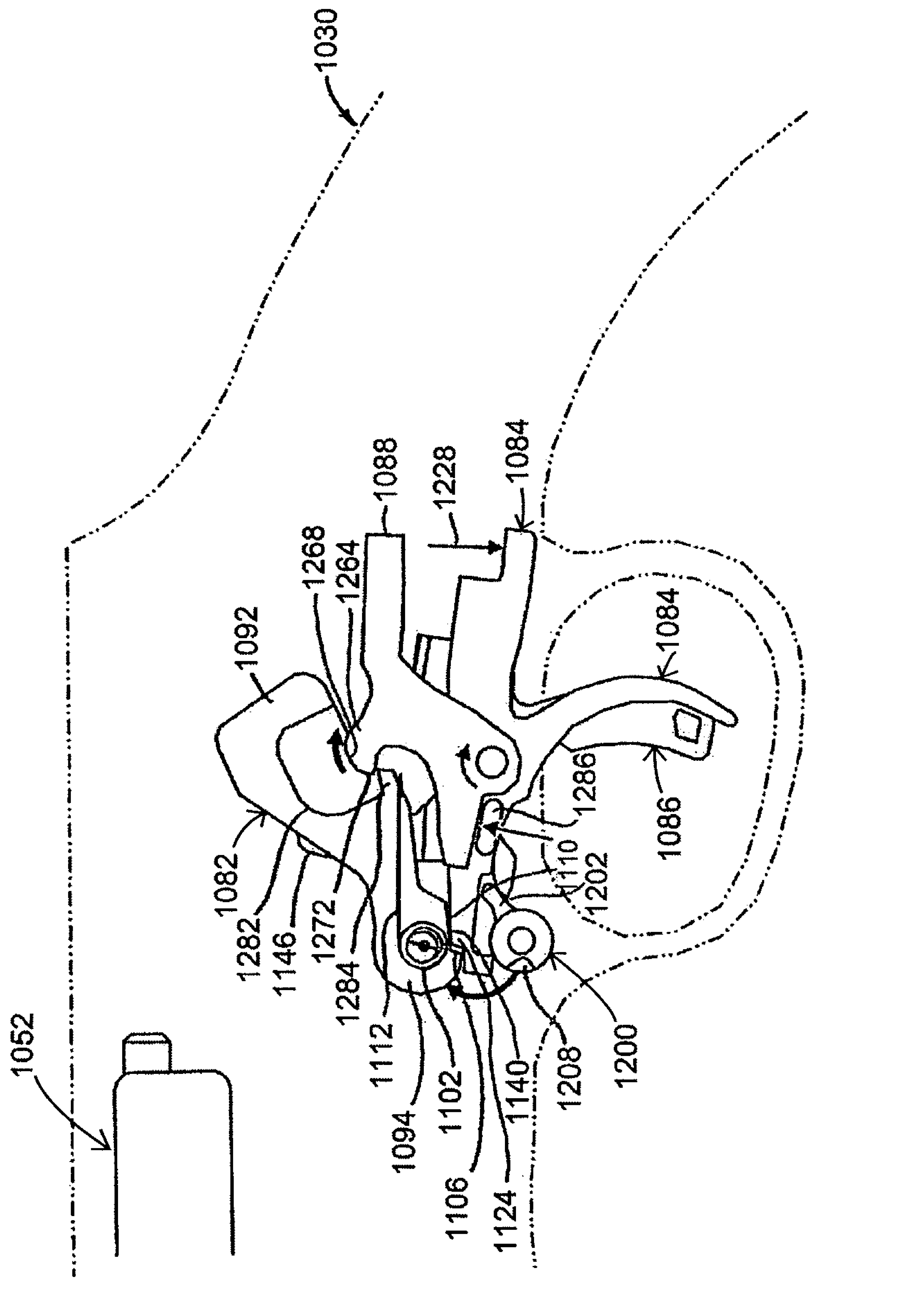

1080
1030
1084
1088
1126
1264
1268
1272
1092
1082
1282
1284
1296
1292
1268
1266
1102
1052
1260
1081

FIG. 73


1030
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1268
1264
1092
1084
1286
1086
1110
1202
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1282
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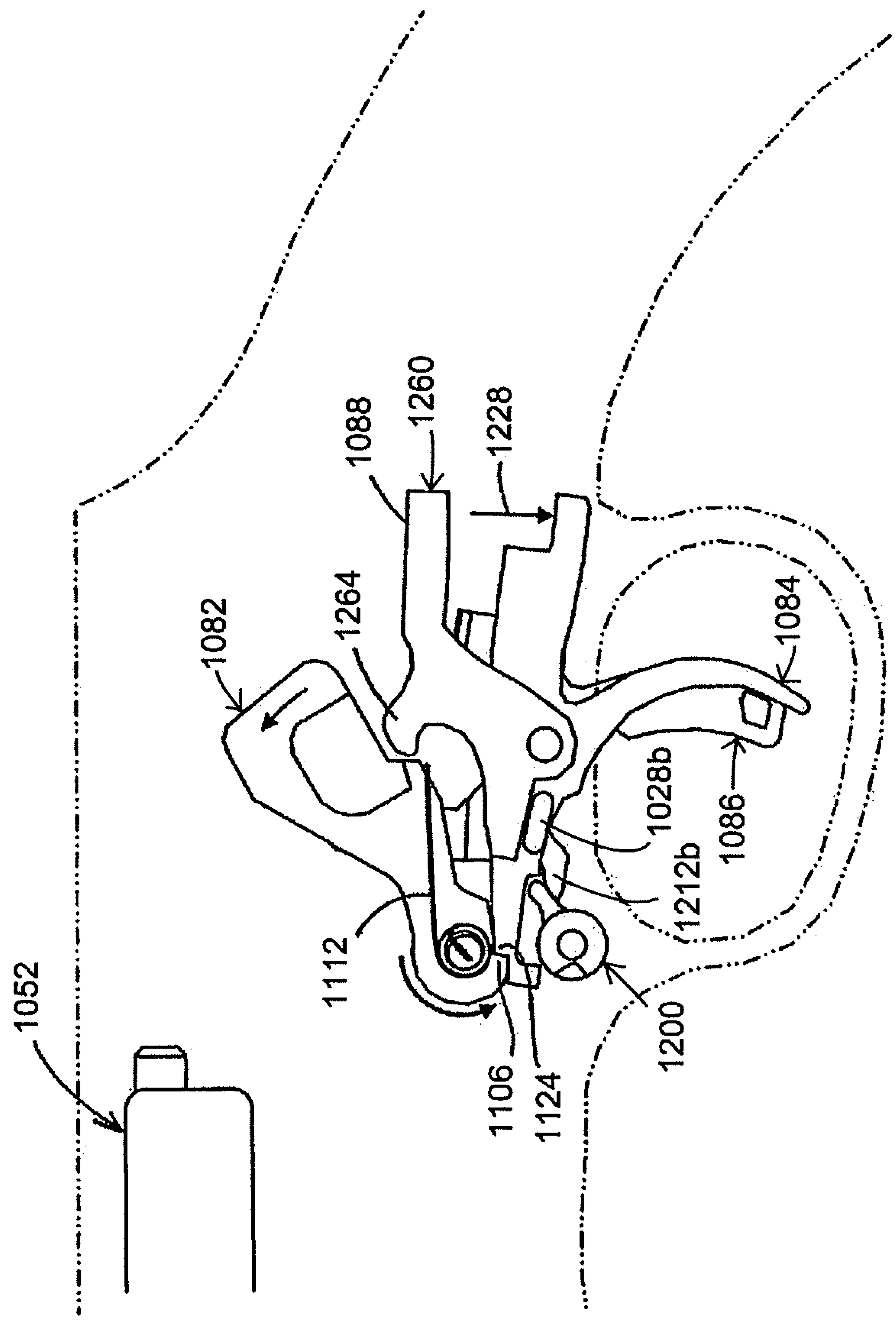
1272
1284
1112
1094
1102
1140
1208
1200
1106
1124
1052

FIG. 74


1088
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1264
1082
1084
1028b
1086
1212b
1112
1200
1052
1106
1124

FIG. 75

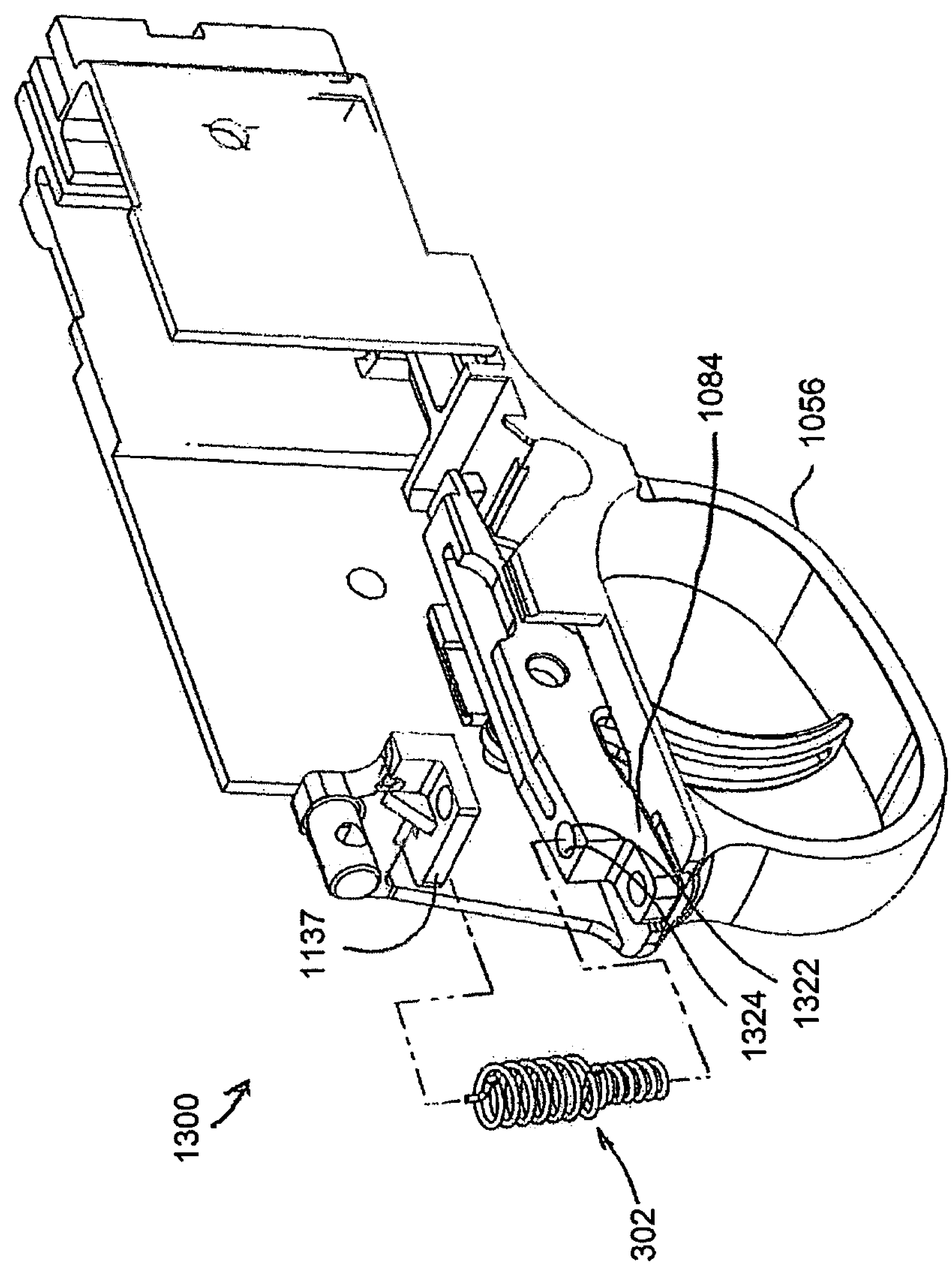
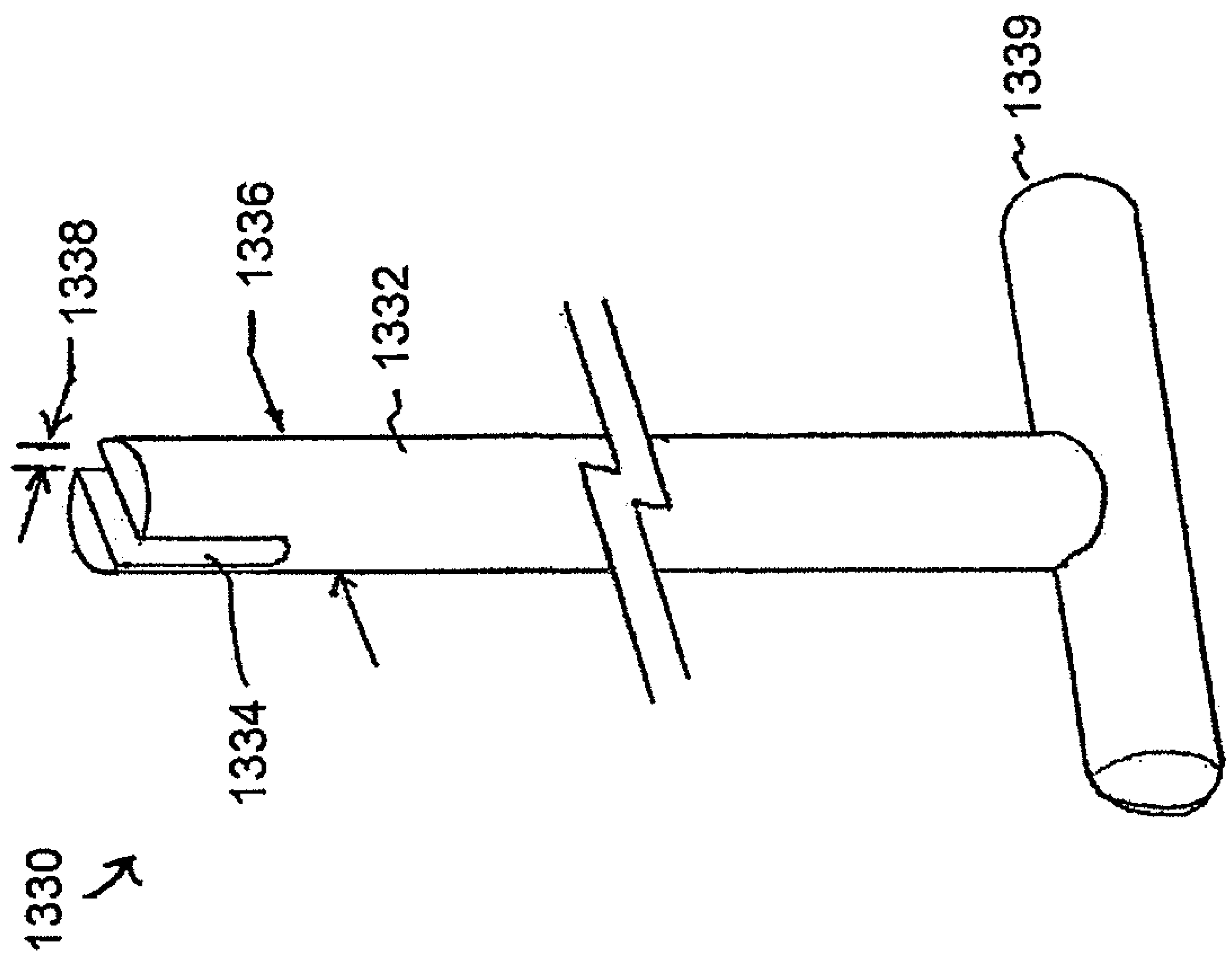


FIG. 76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1336
1332
1339
1334
1330

FIG. 78

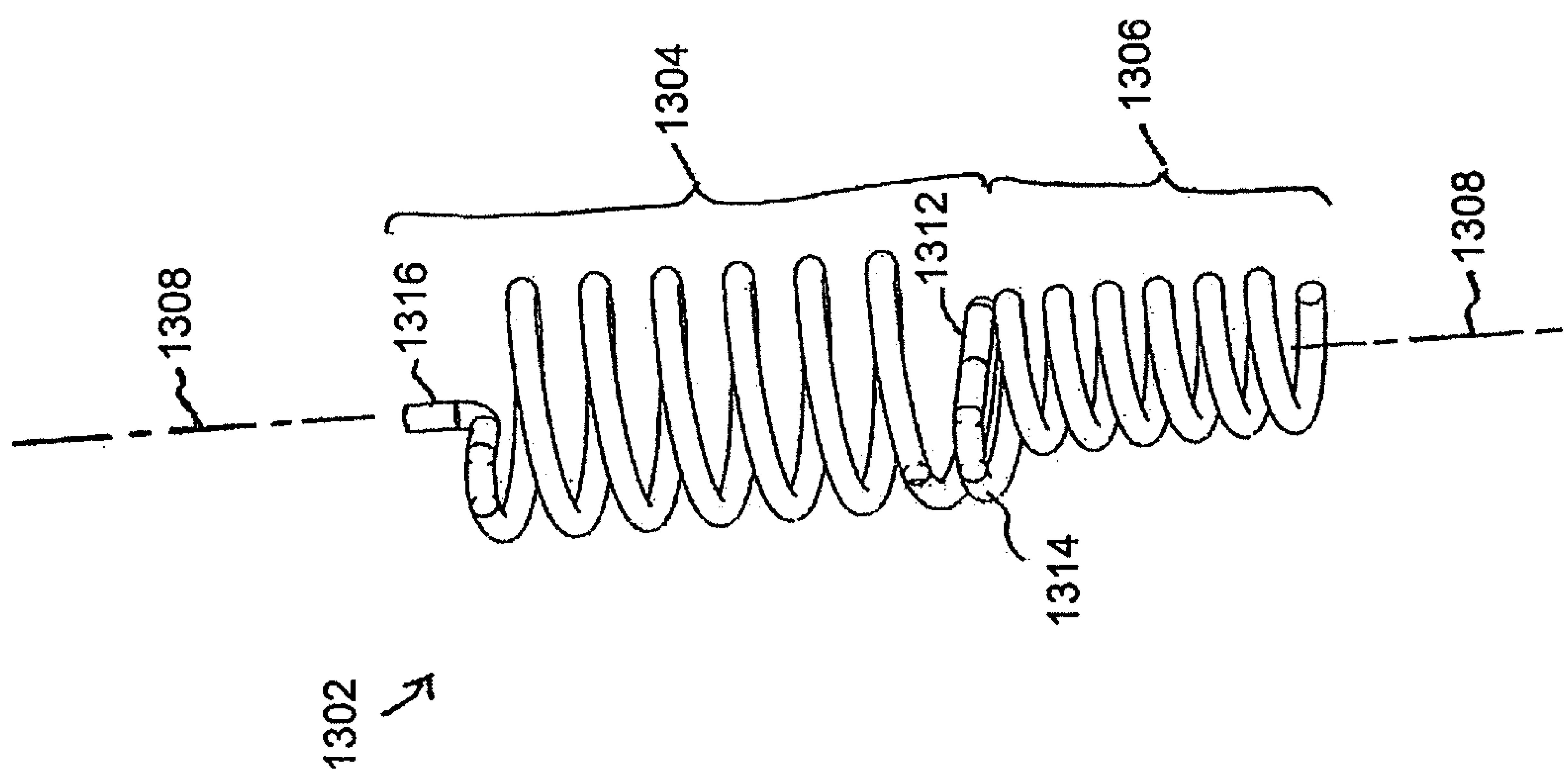

1304
1306
1308
1316
1312
1308
1314
1302

FIG. 77

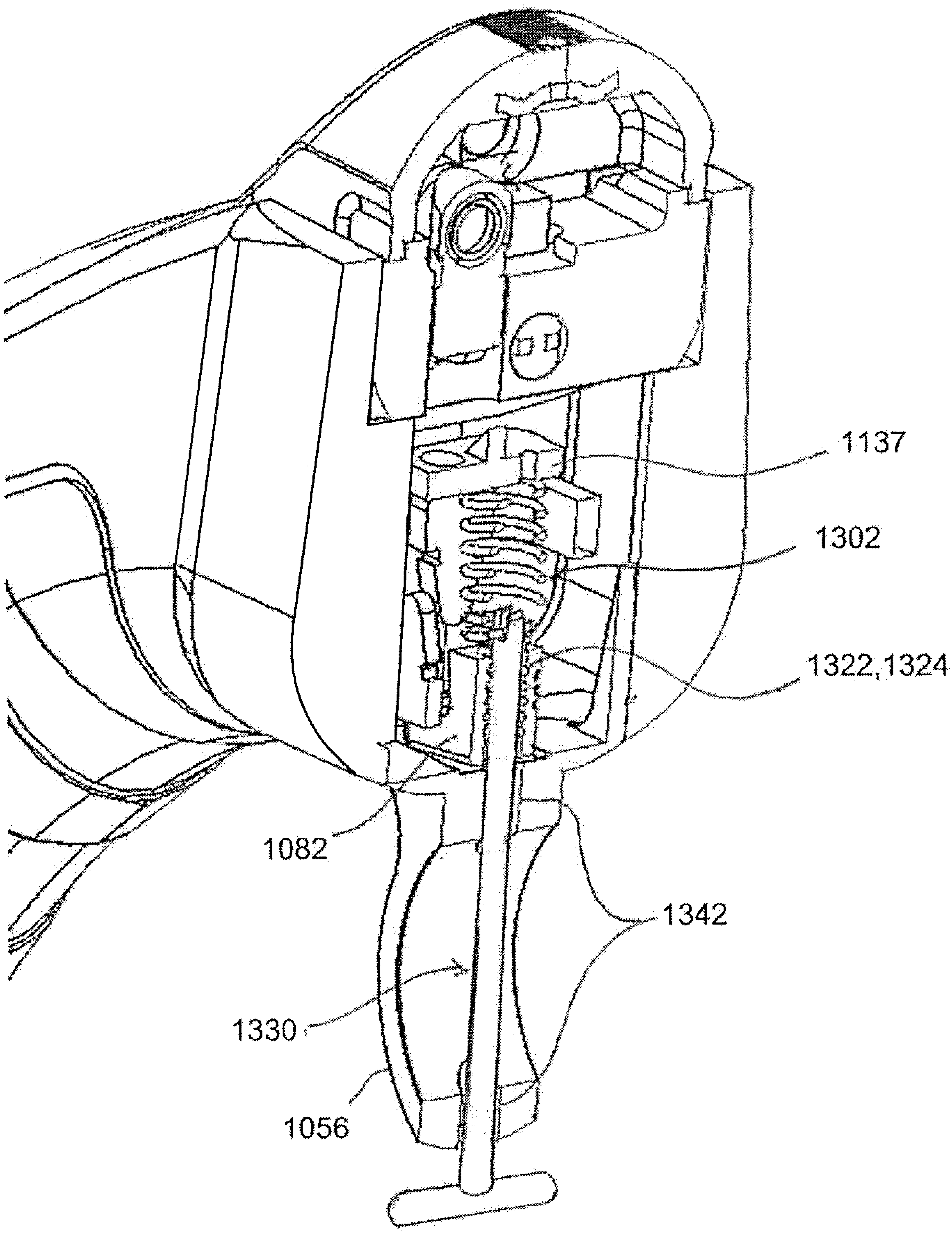


FIG. 79

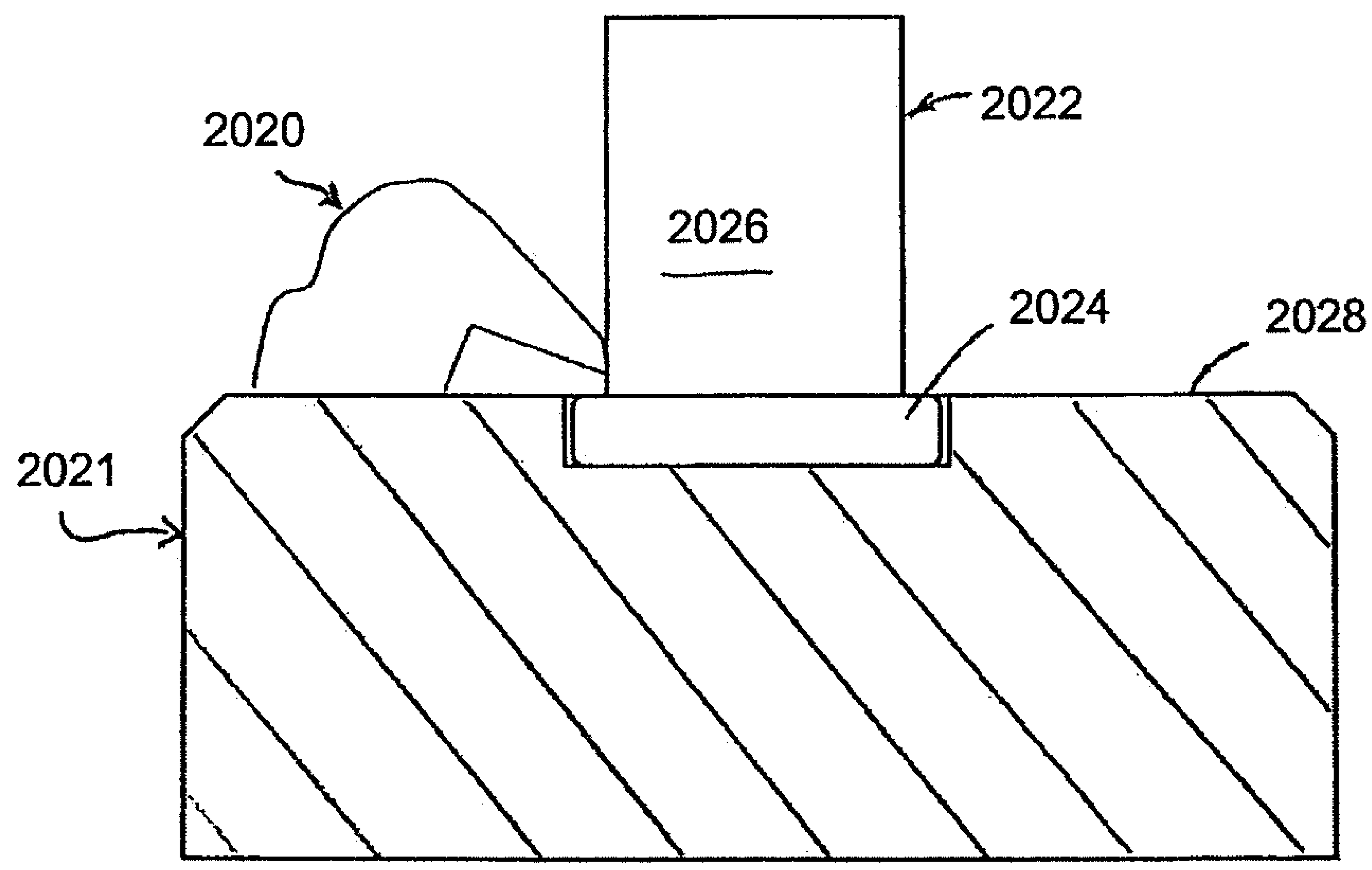


FIG. 80
PRIOR ART

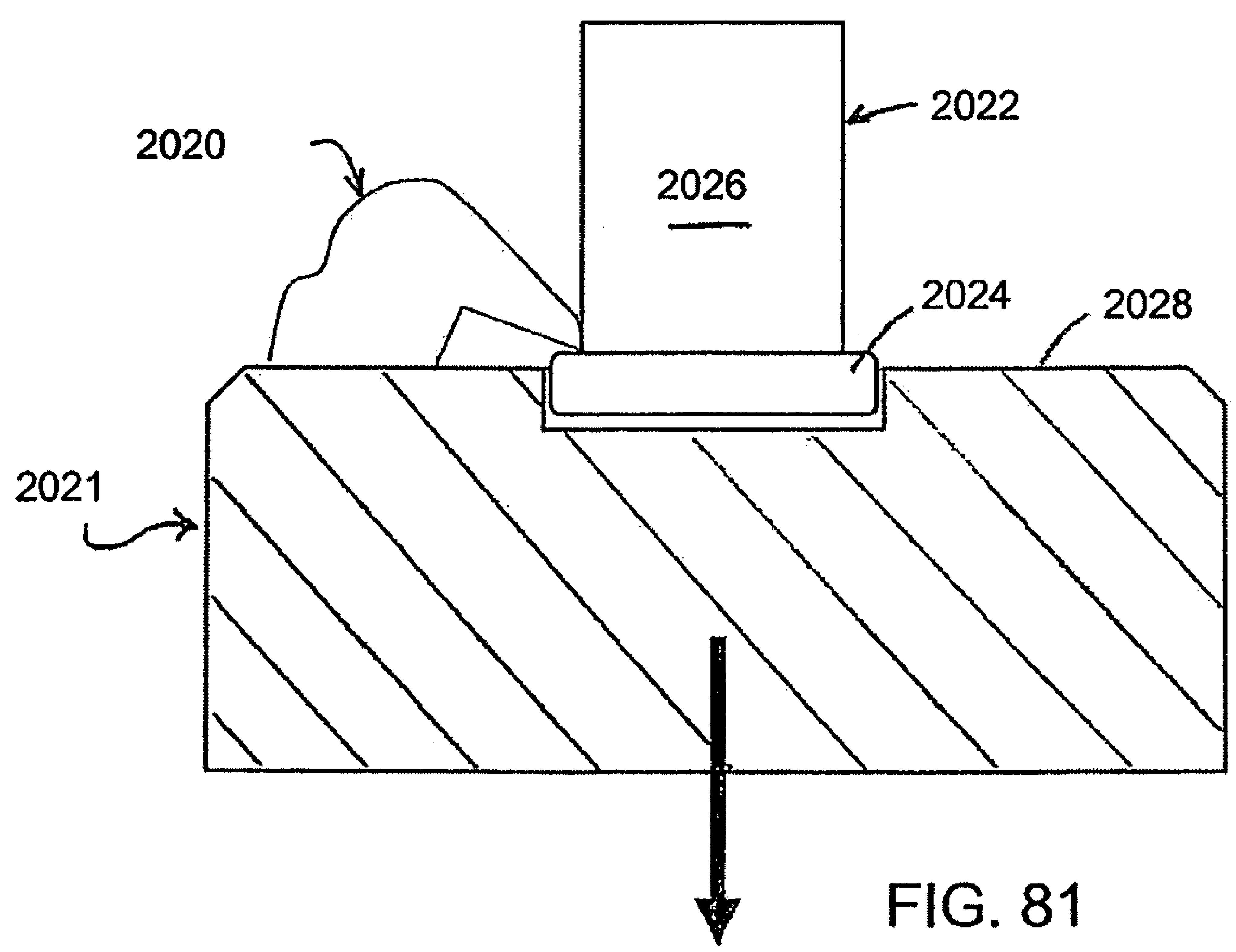


FIG. 81

PRIOR ART

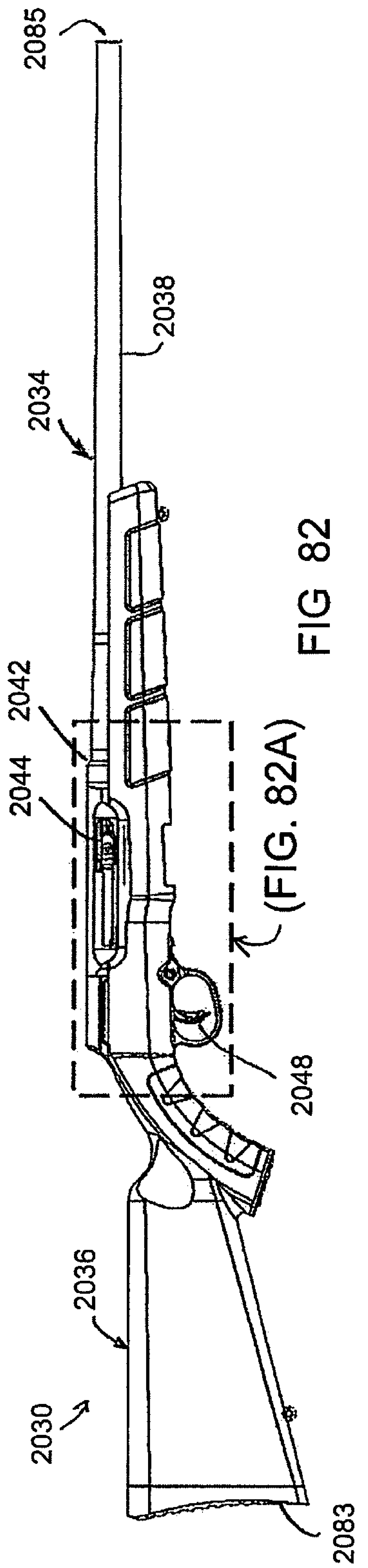


FIG 82

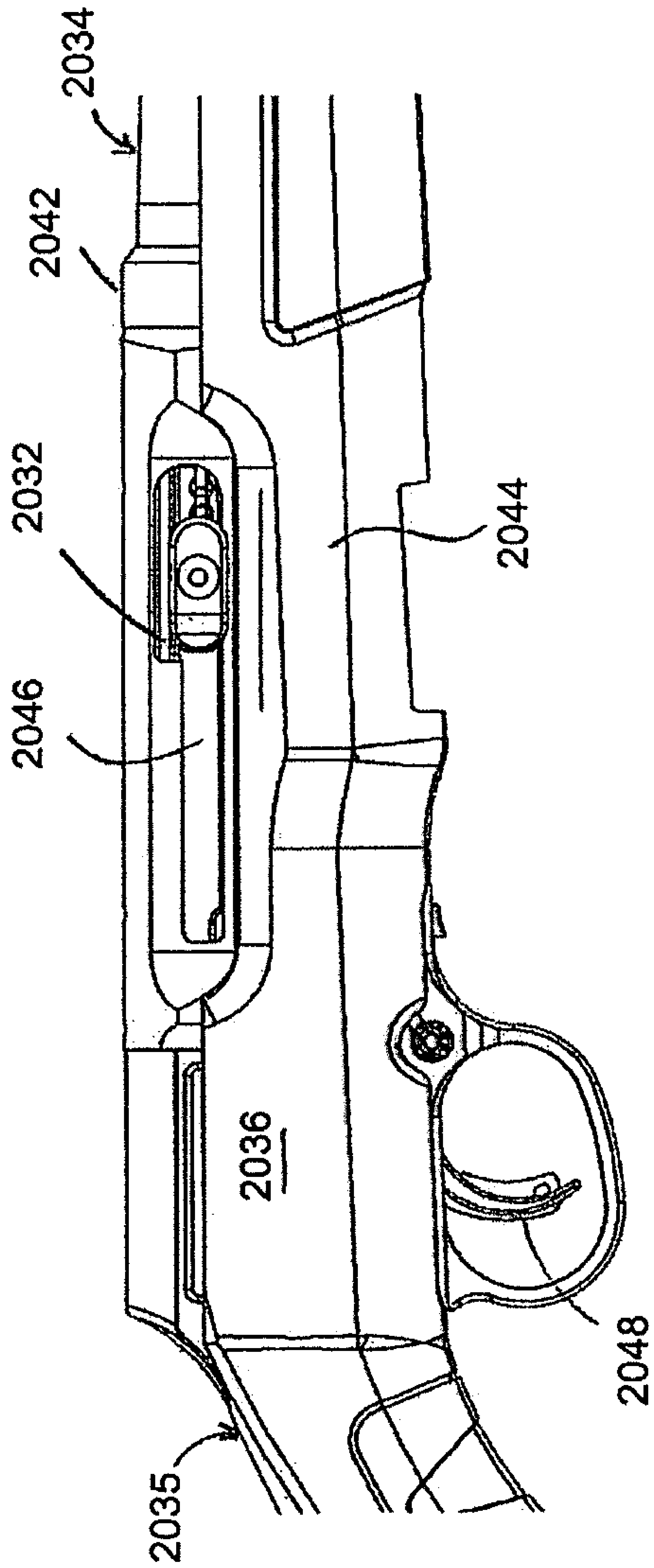
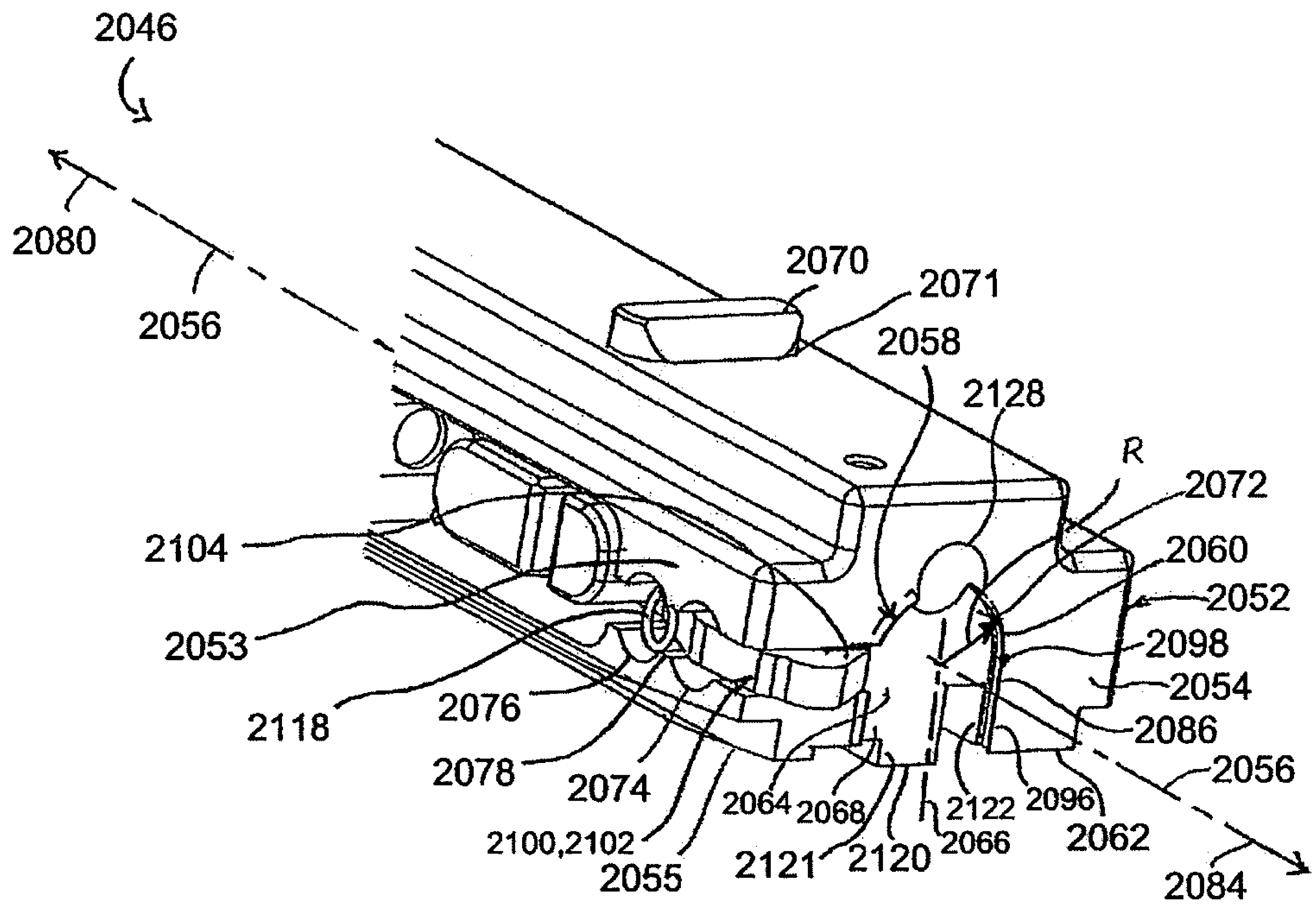


FIG. 82A


2046
2080
2056
2070
2071
2058
2128
R
2072
2060
2052
2098
2054
2086
2056
2104
2053
2118
2076
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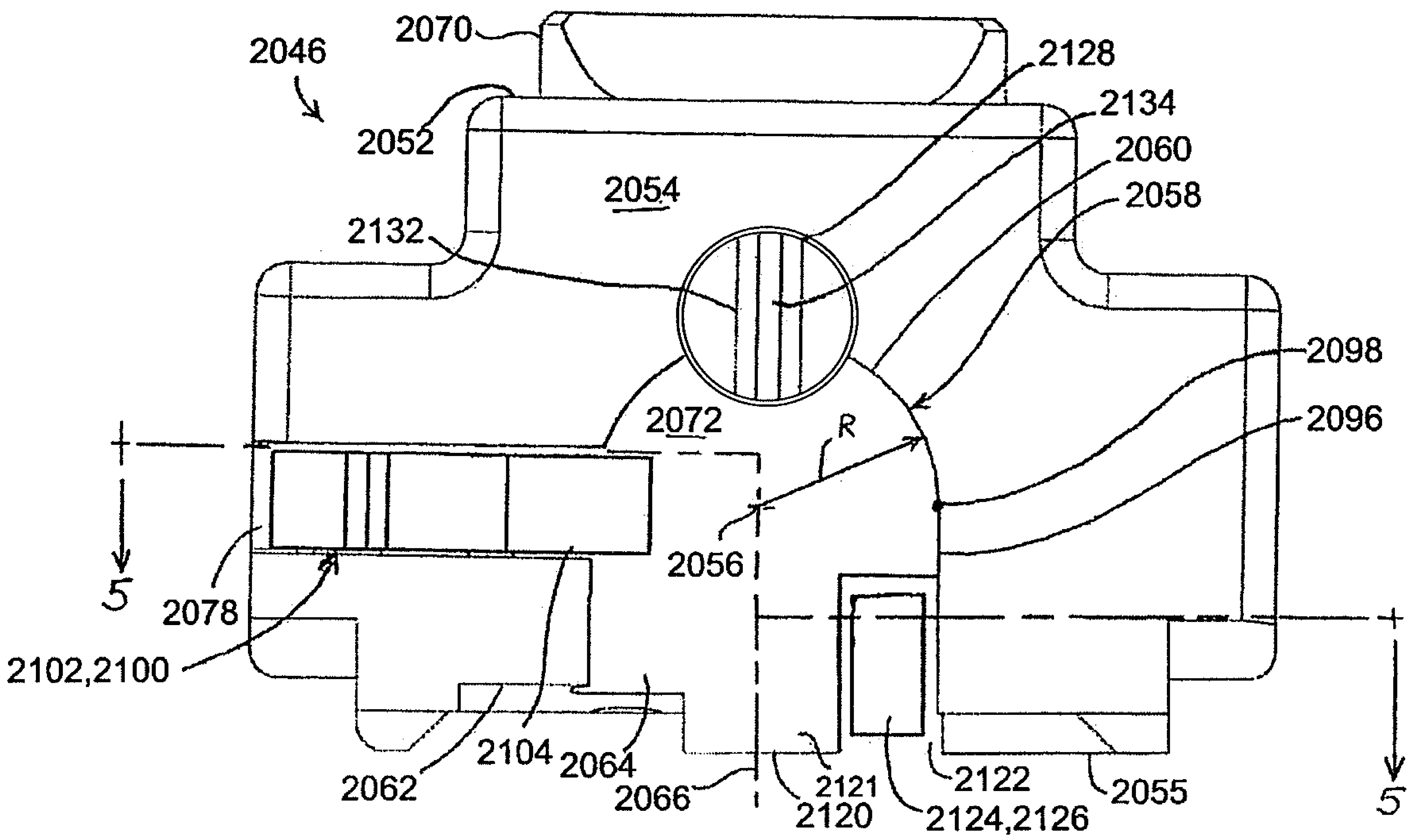
2074
2100,2102
2055
2064
2068
2121
2120
2122
2066
2096
2062
2084

FIG. 83


2070
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2054
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2060
2058
2132
2072
R
2098
2096
2056
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5
2102,2100
2062
2104
2064
2066
2121
2120
2122
2124,2126
2055
5

FIG. 84

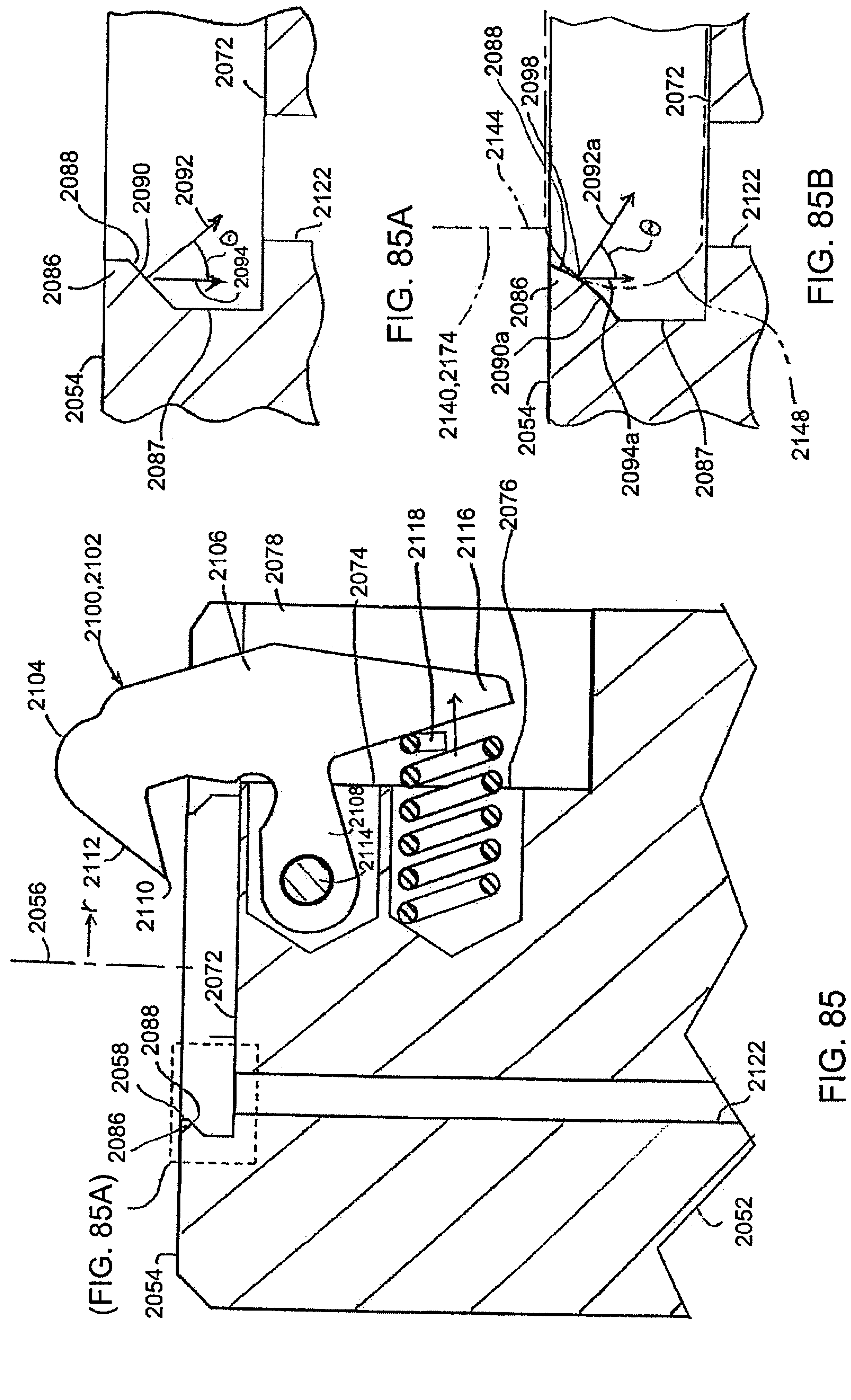


FIG. 85A

FIG. 85B

FIG. 85

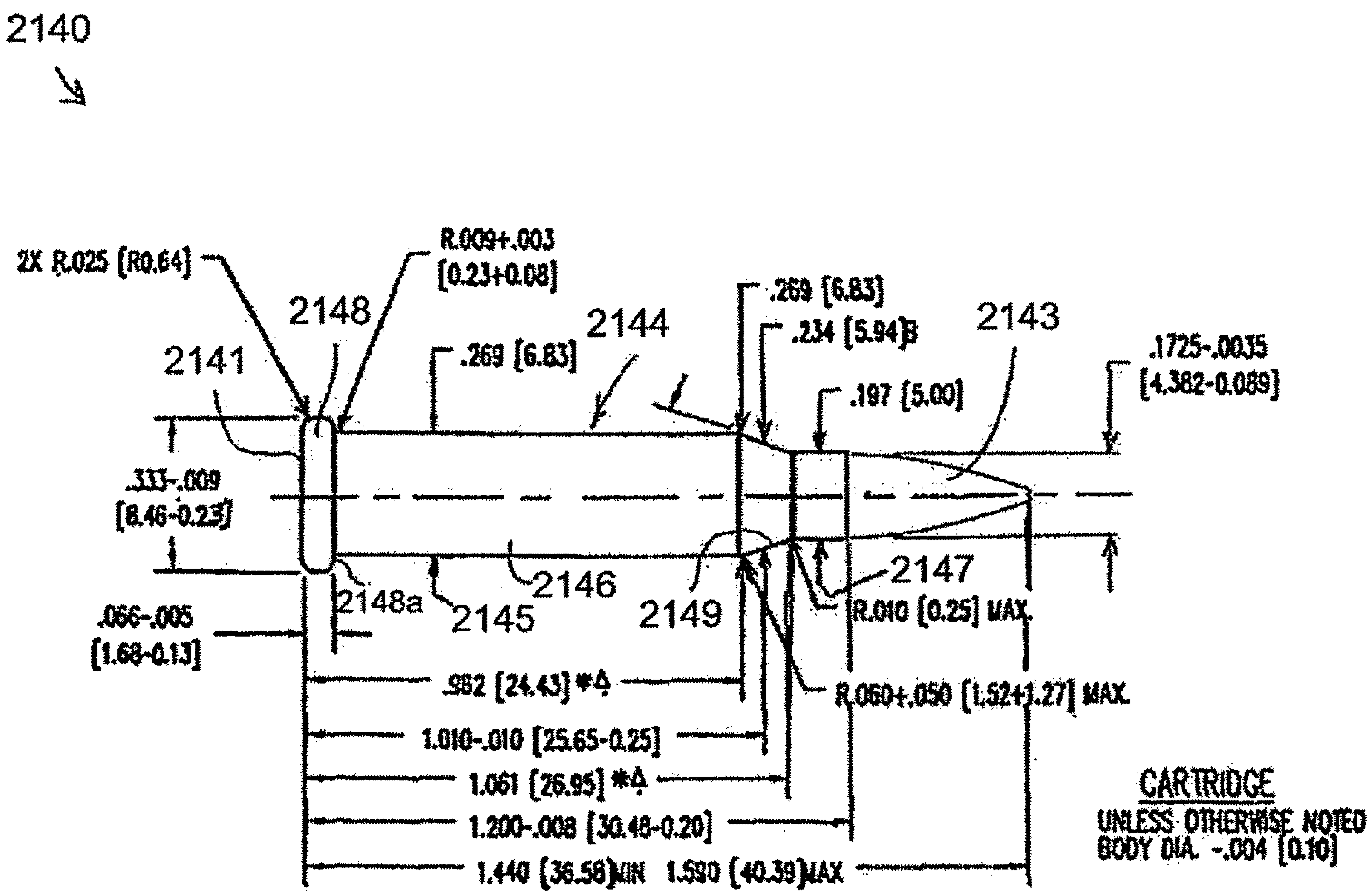


FIG. 85C

PRIOR ART

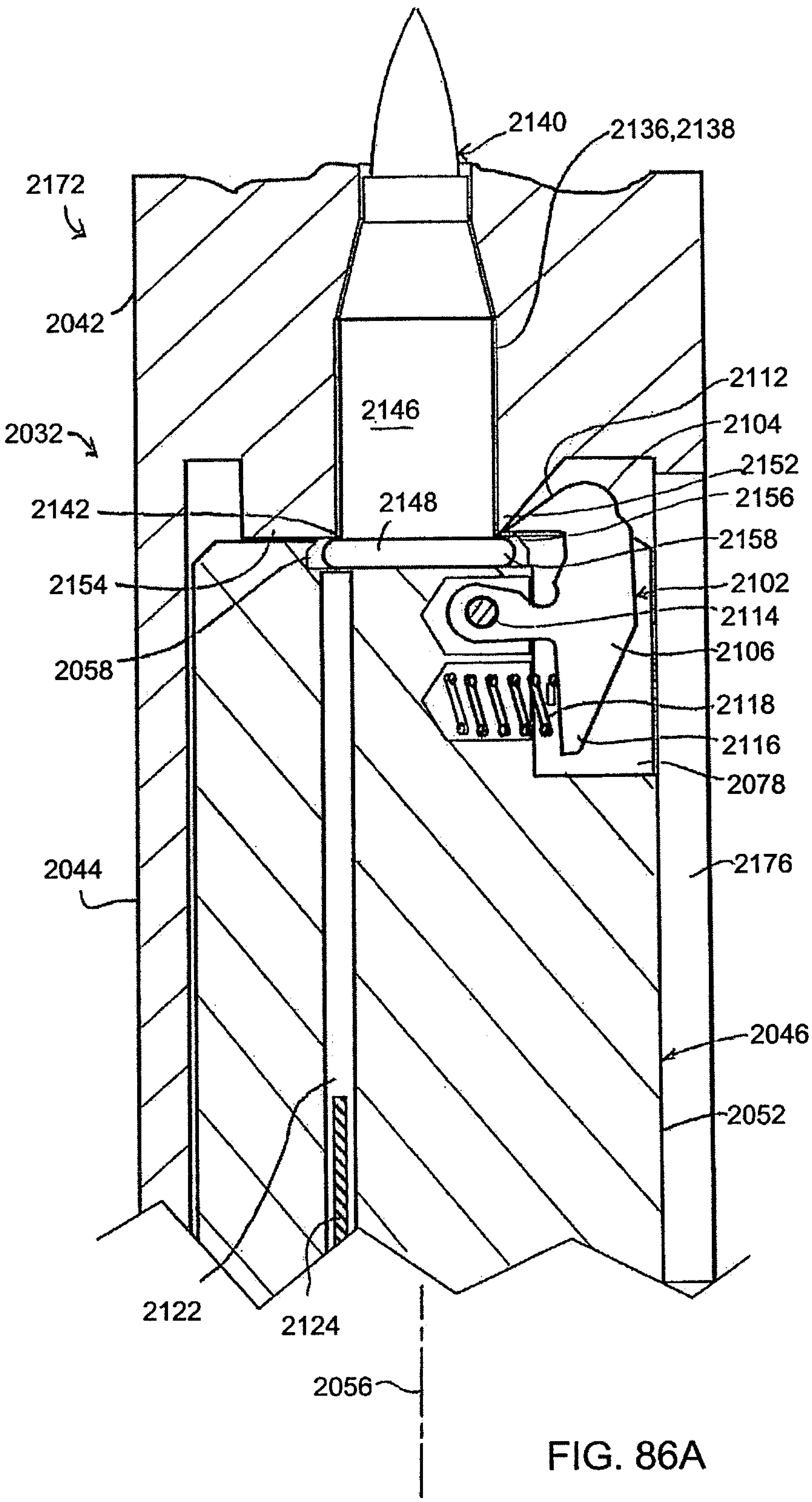
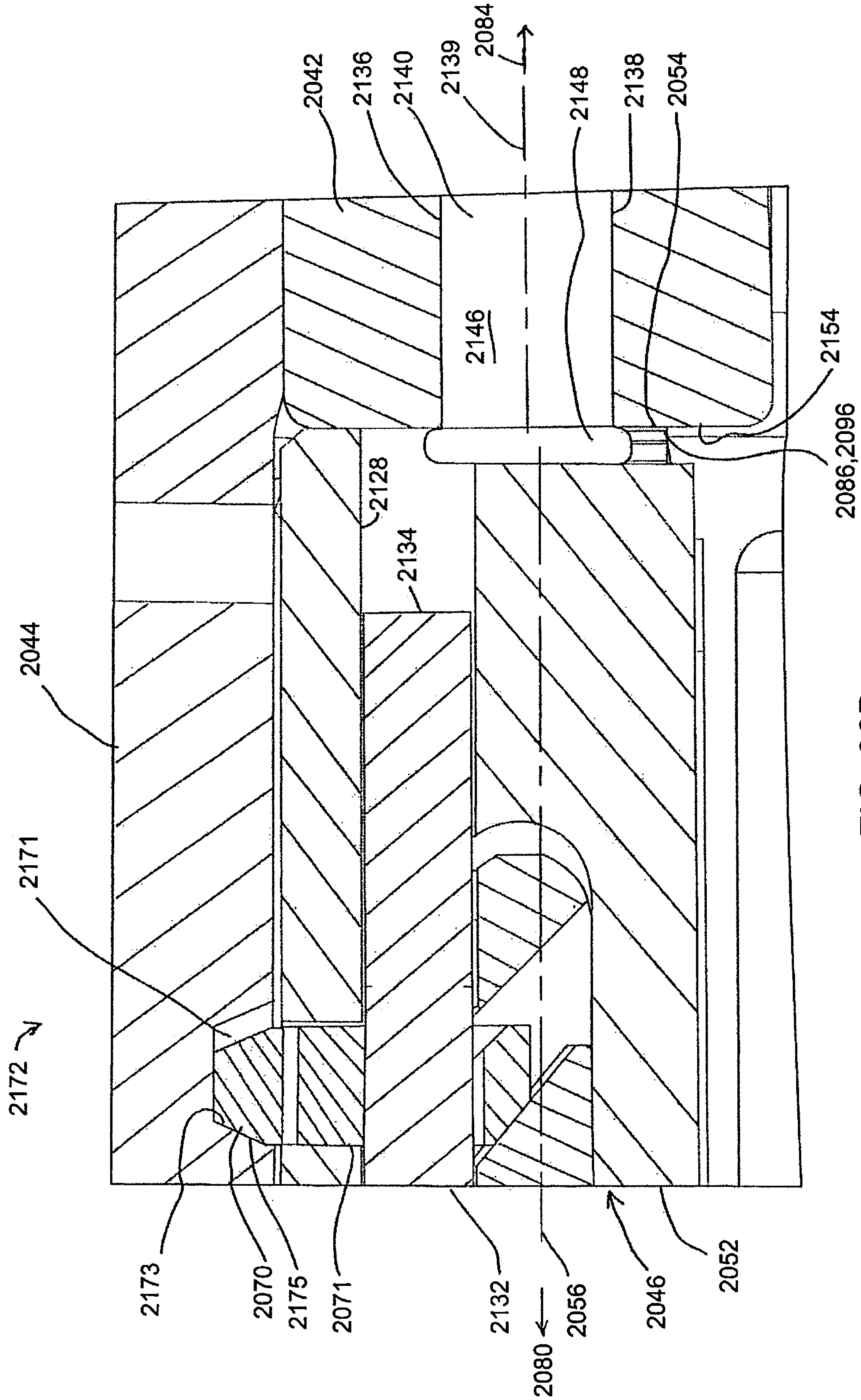

2140
2136,2138
2172
2042
2146
2112
2104
2152
2032
2156
2142
2148
2158
2154
2102
2114
2106
2058
2118
2116
2078
2044
2176
2046
2052
2122
2124
2056

FIG. 86A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2146
2154
2086,2096
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2134
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2171
2172
2173
2070
2175
2071
2132
2056
2046
2052
2080

FIG. 86B

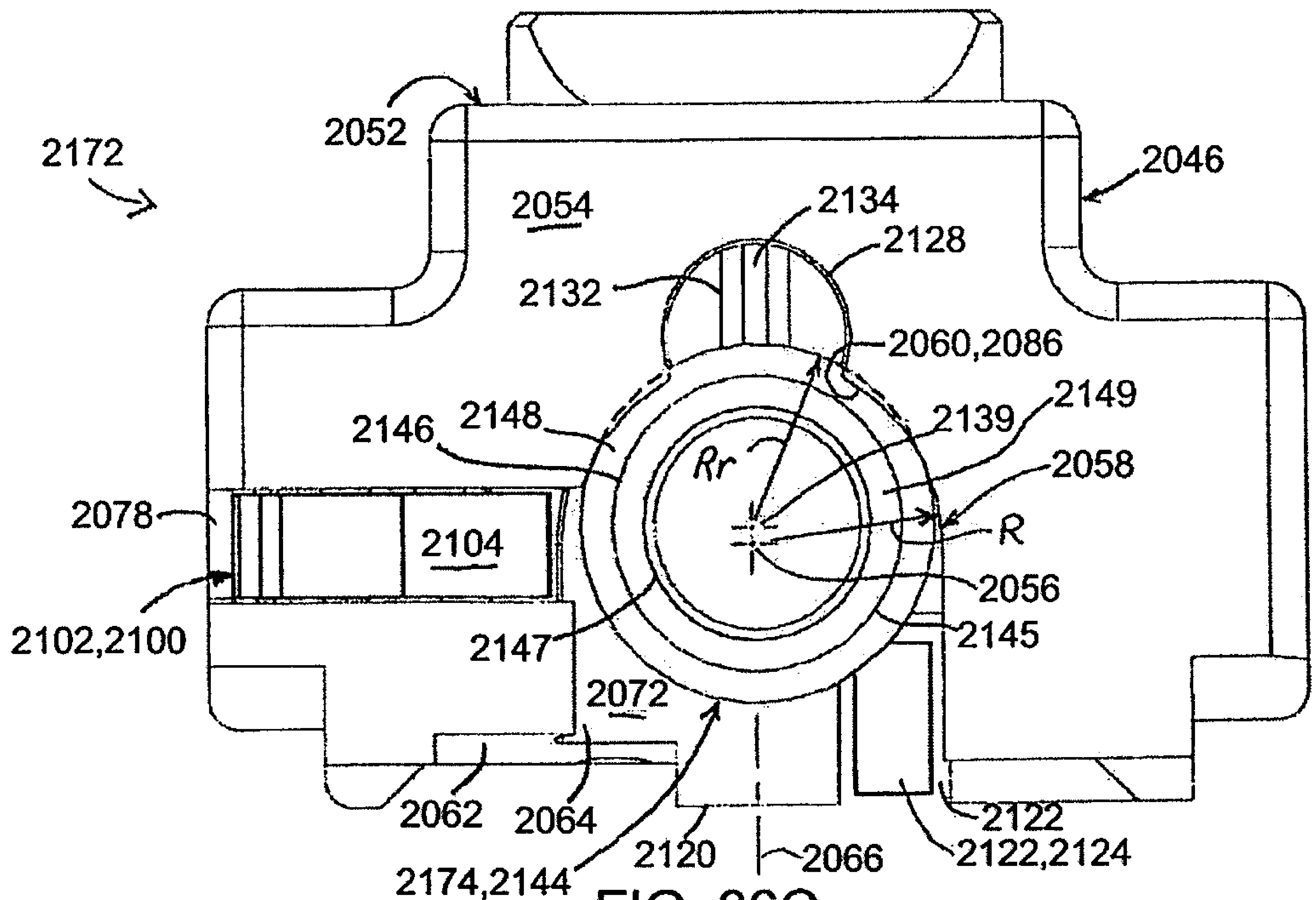


FIG. 86C

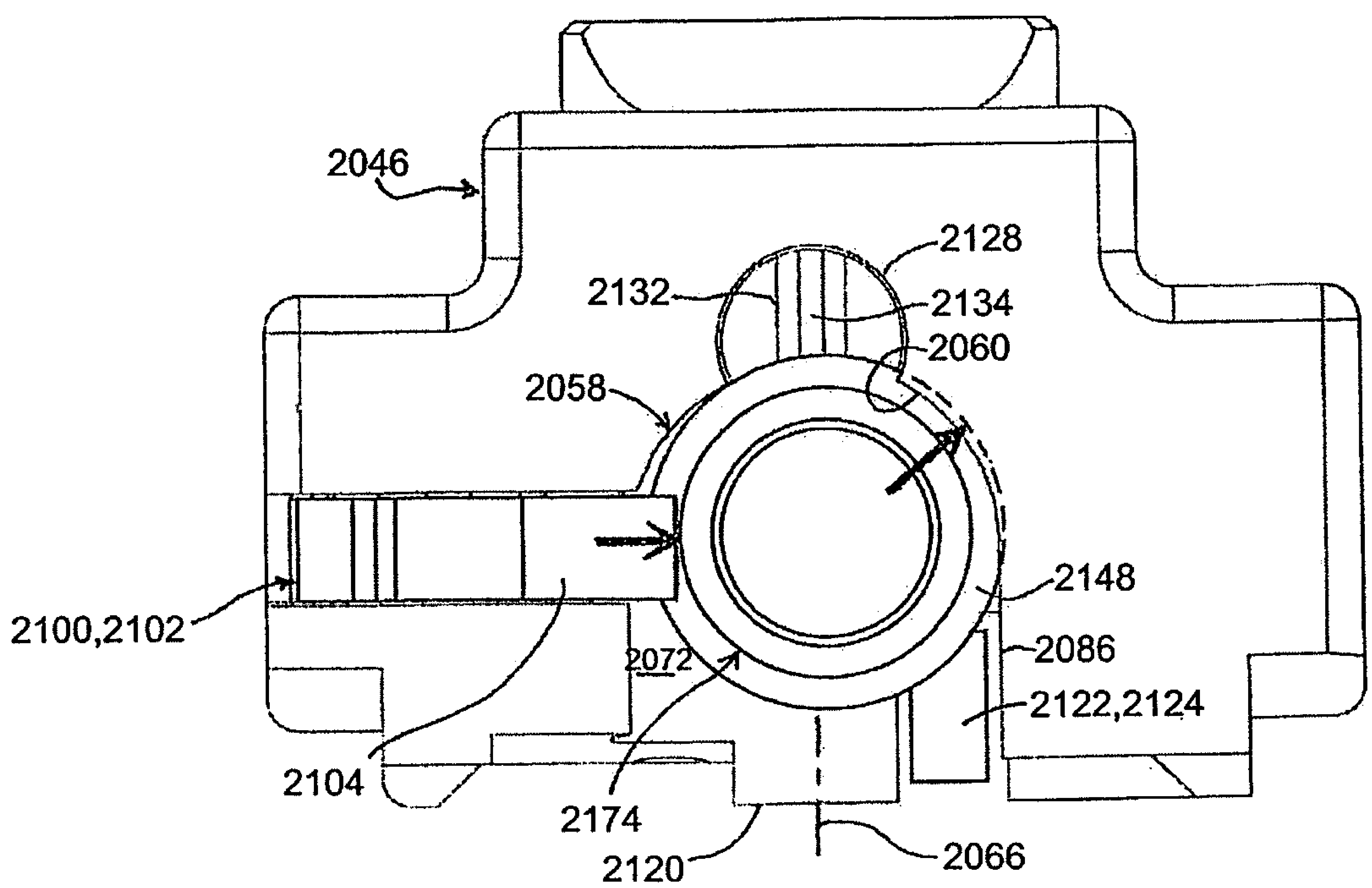
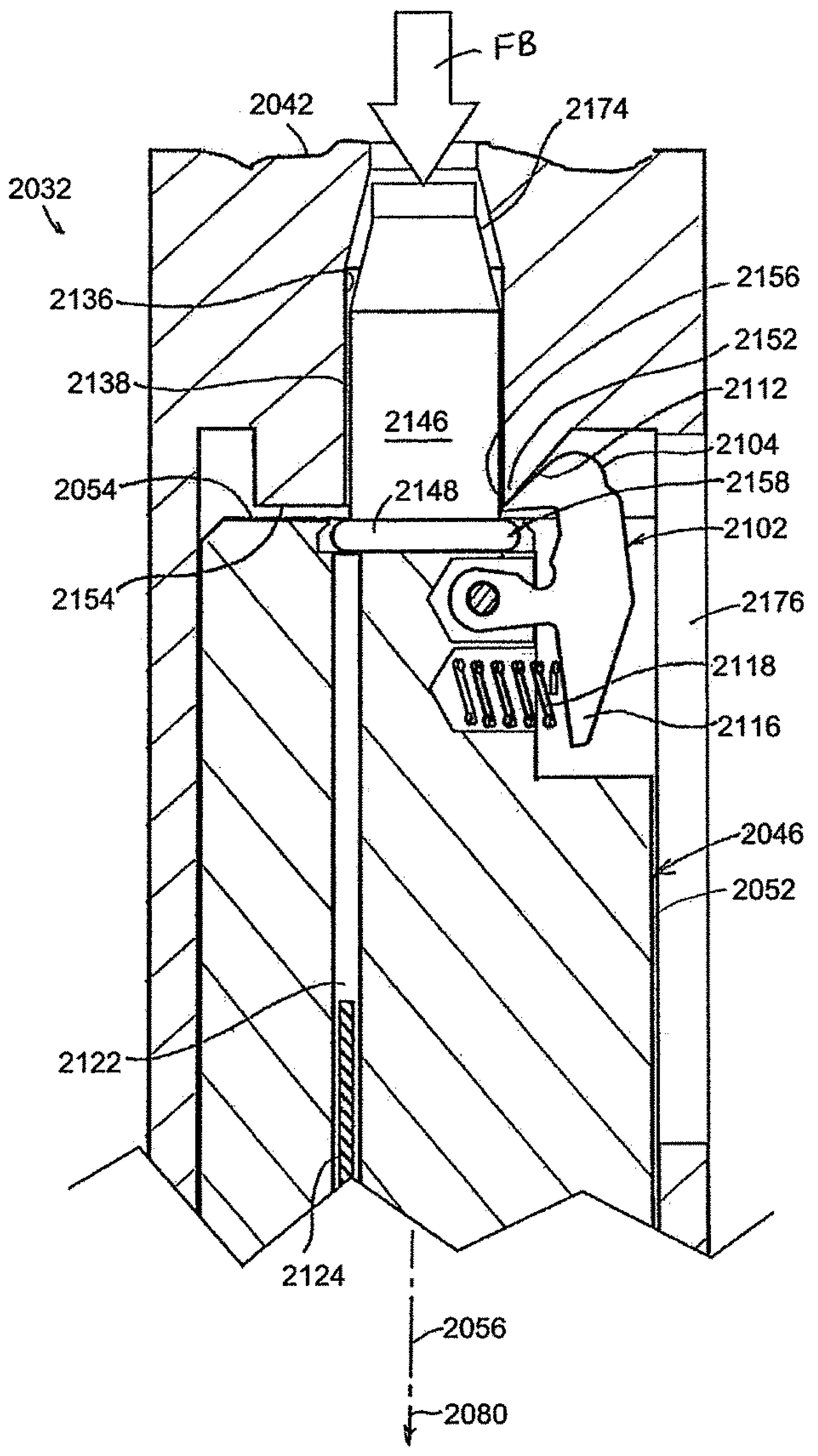


FIG. 86G



FB
2042
2174
2032
2136
2156
2152
2138
2112
2146
2104
2054
2148
2158
2102
2154
2176
2118
2116
2046
2052
2122
2124
2056
2080

FIG. 86D

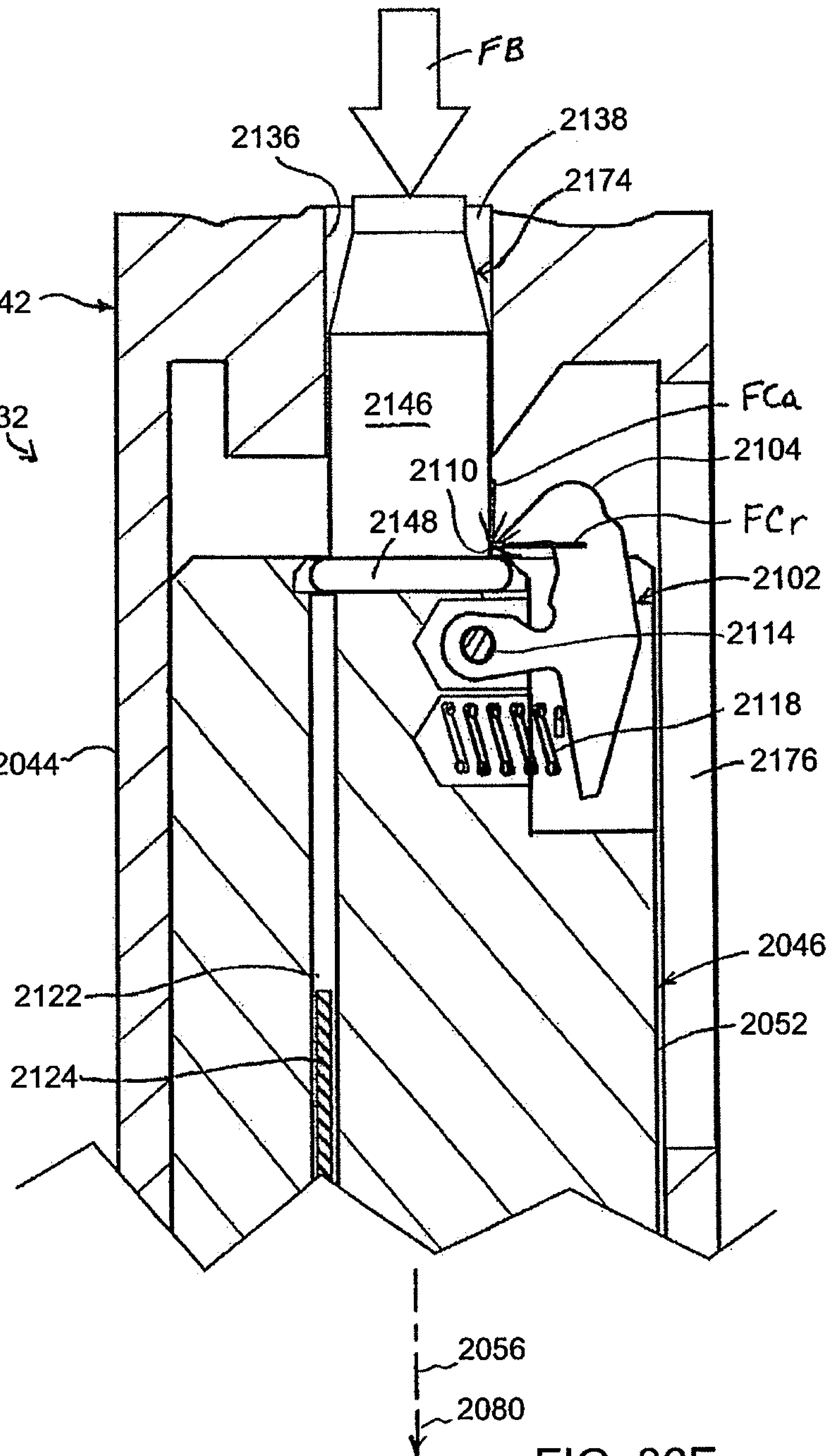

FB
2136
2138
2174
2042
2032
2146
FCa
2104
2110
2148
FCr
2102
2114
2118
2044
2176
2046
2122
2052
2124
2056
2080

FIG. 86E

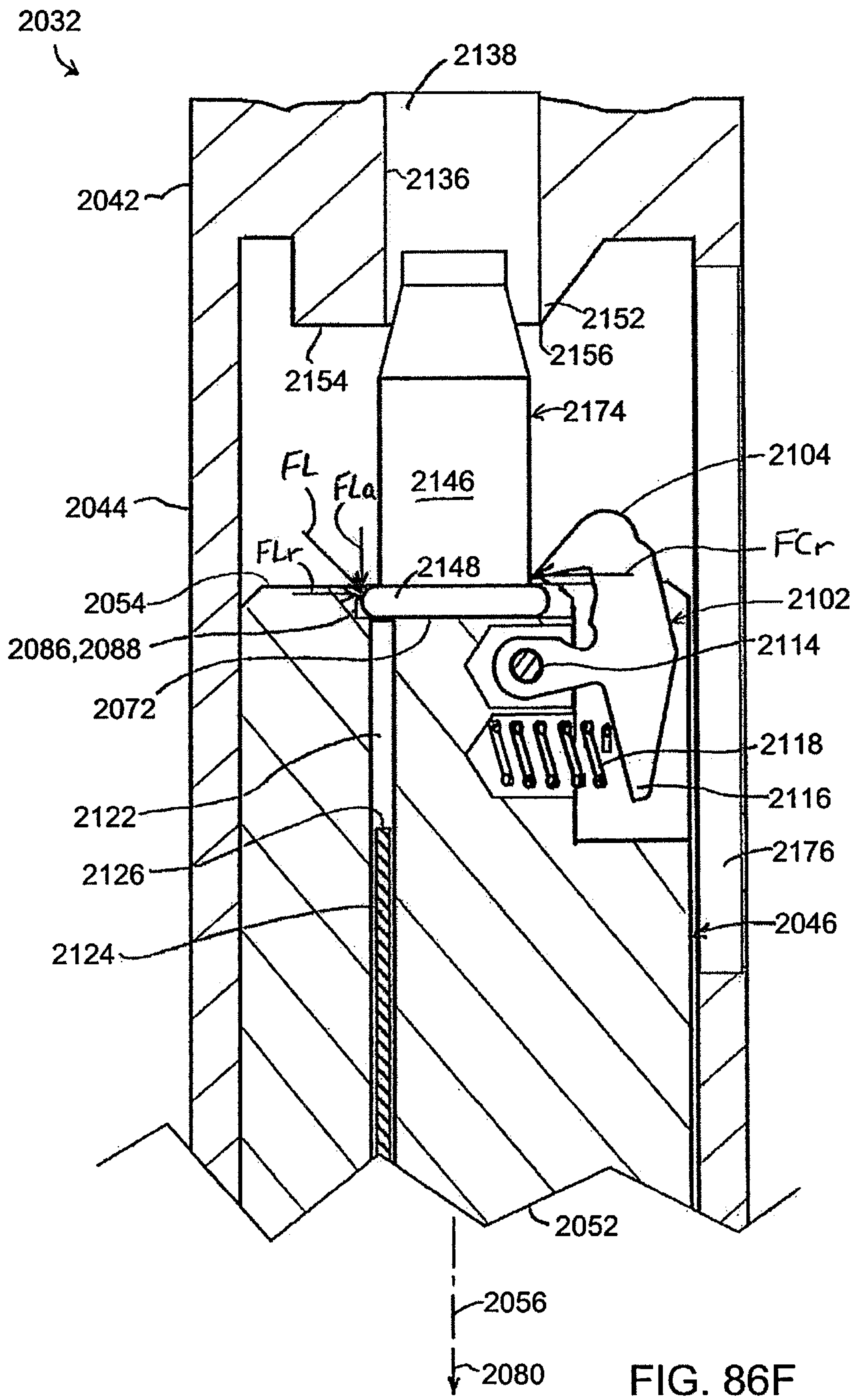


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FIG. 86F

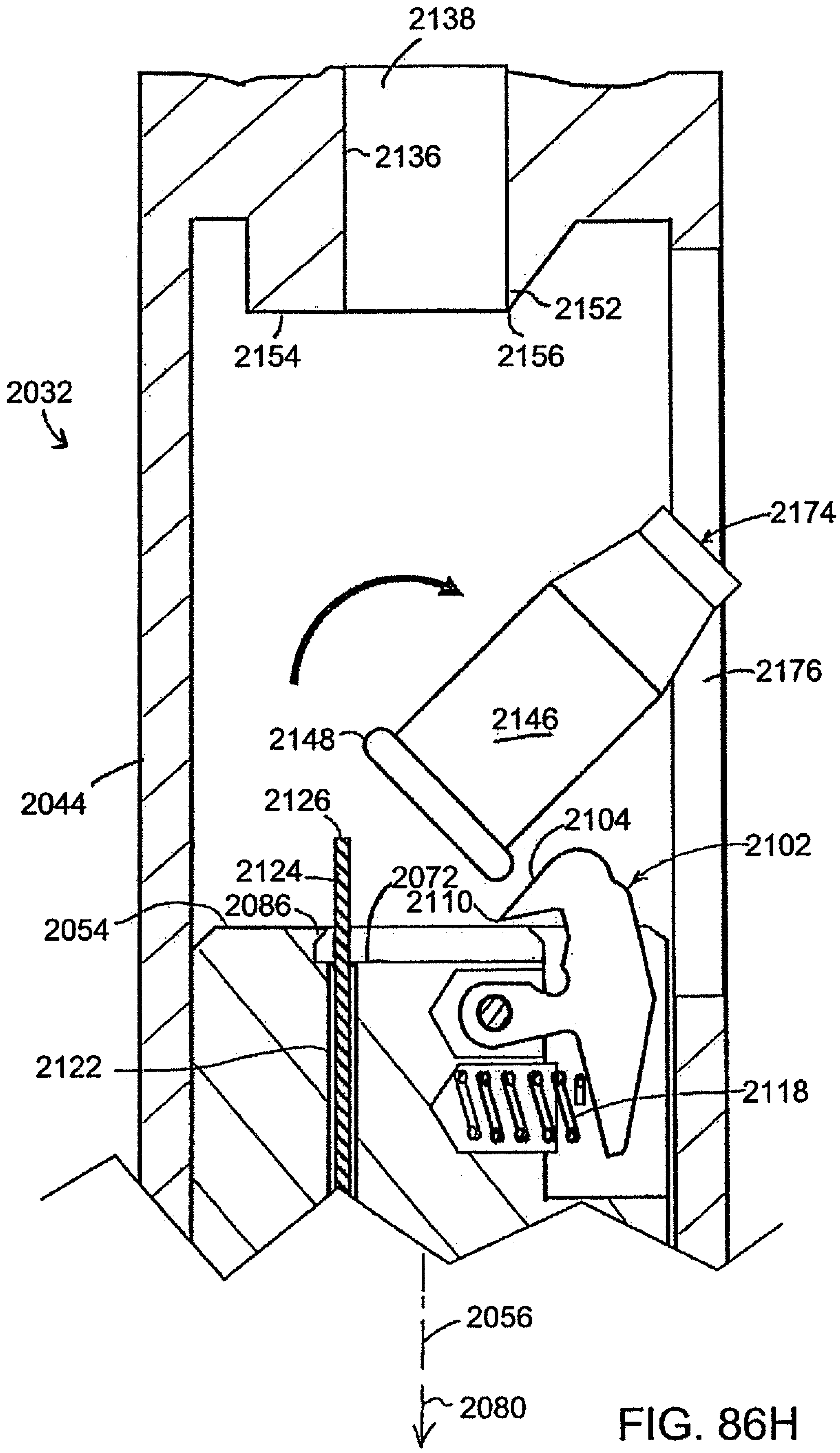
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FIG. 86H

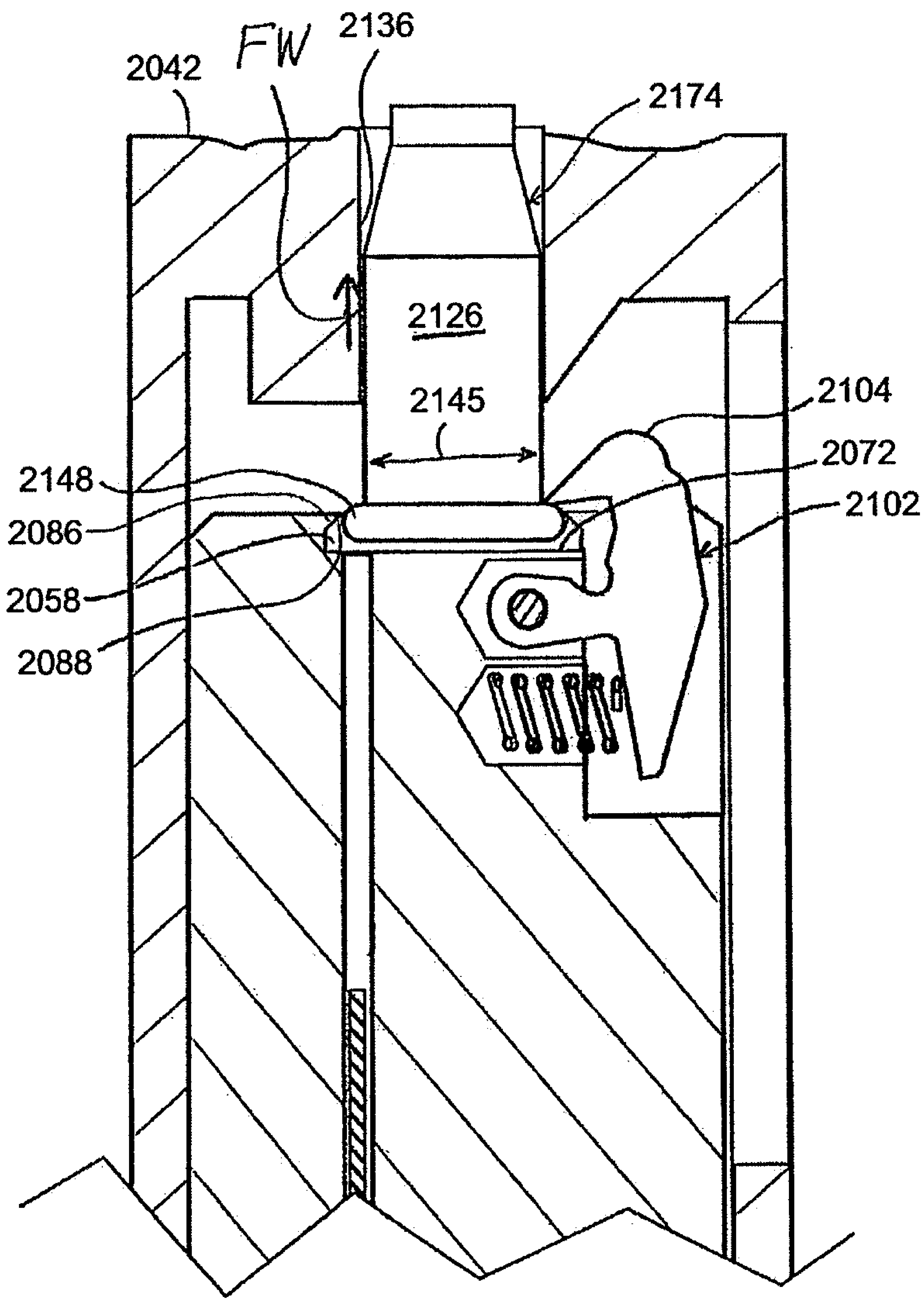
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FIG. 87A

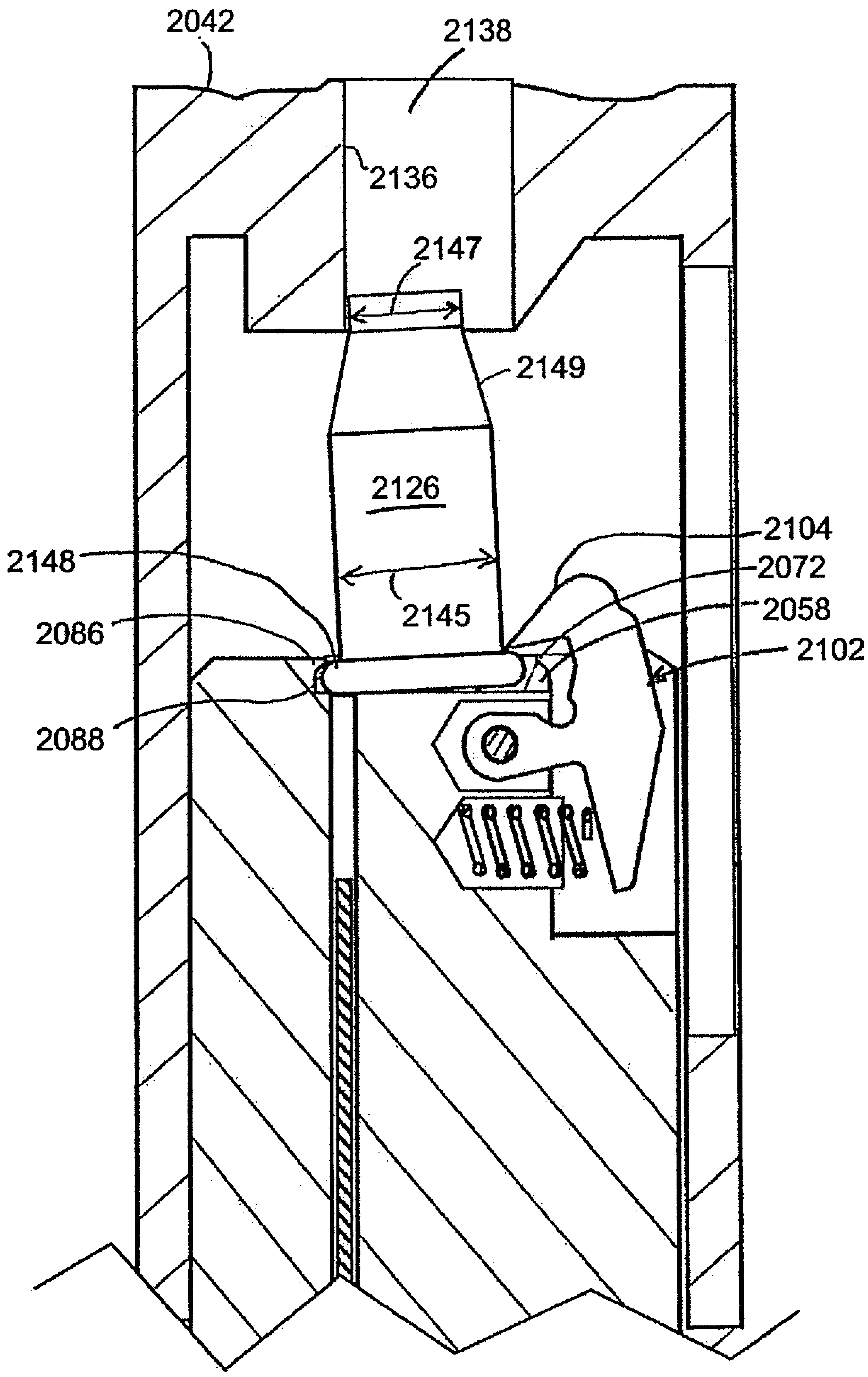
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FIG. 87B

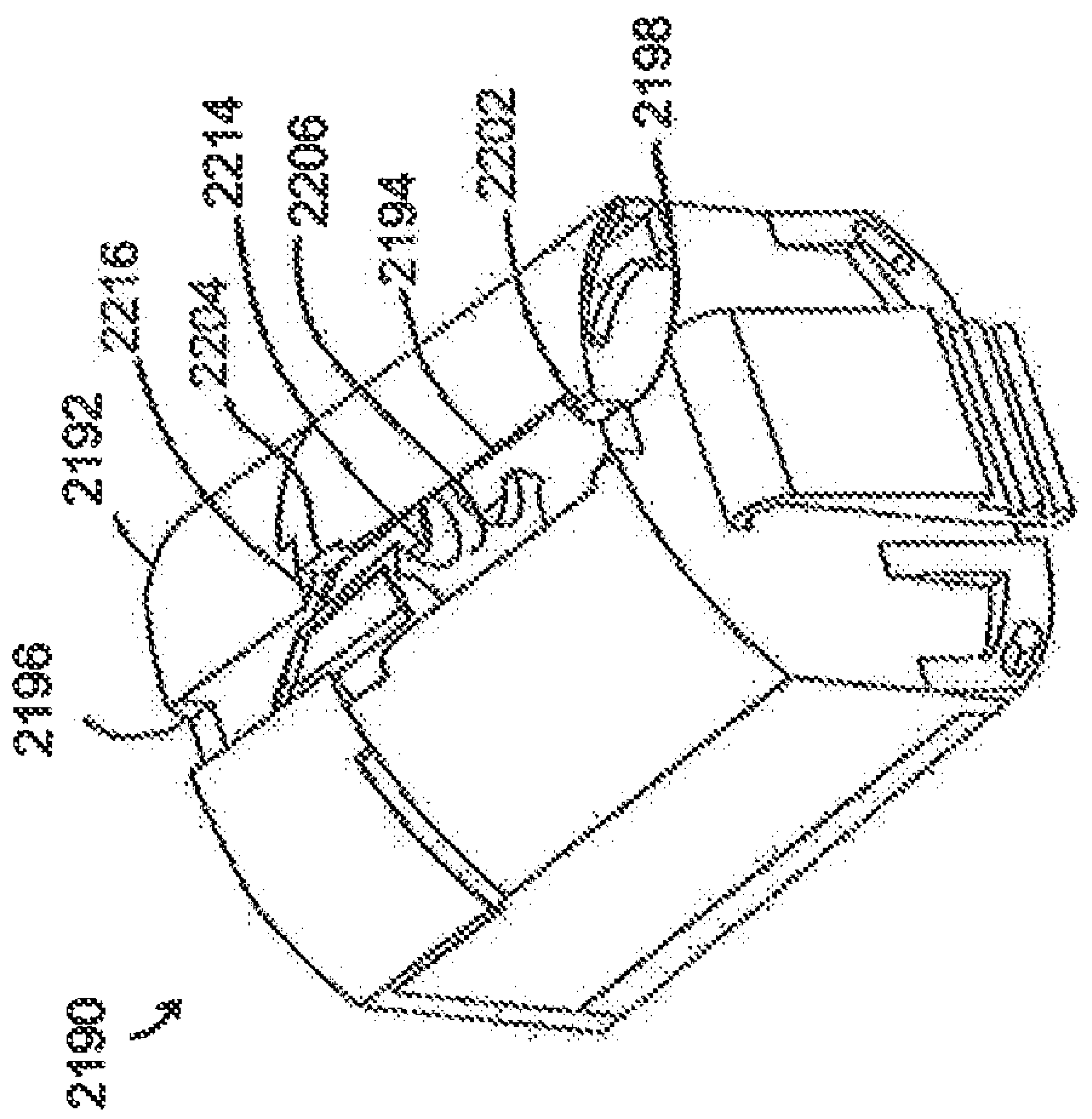

FIG. 88A

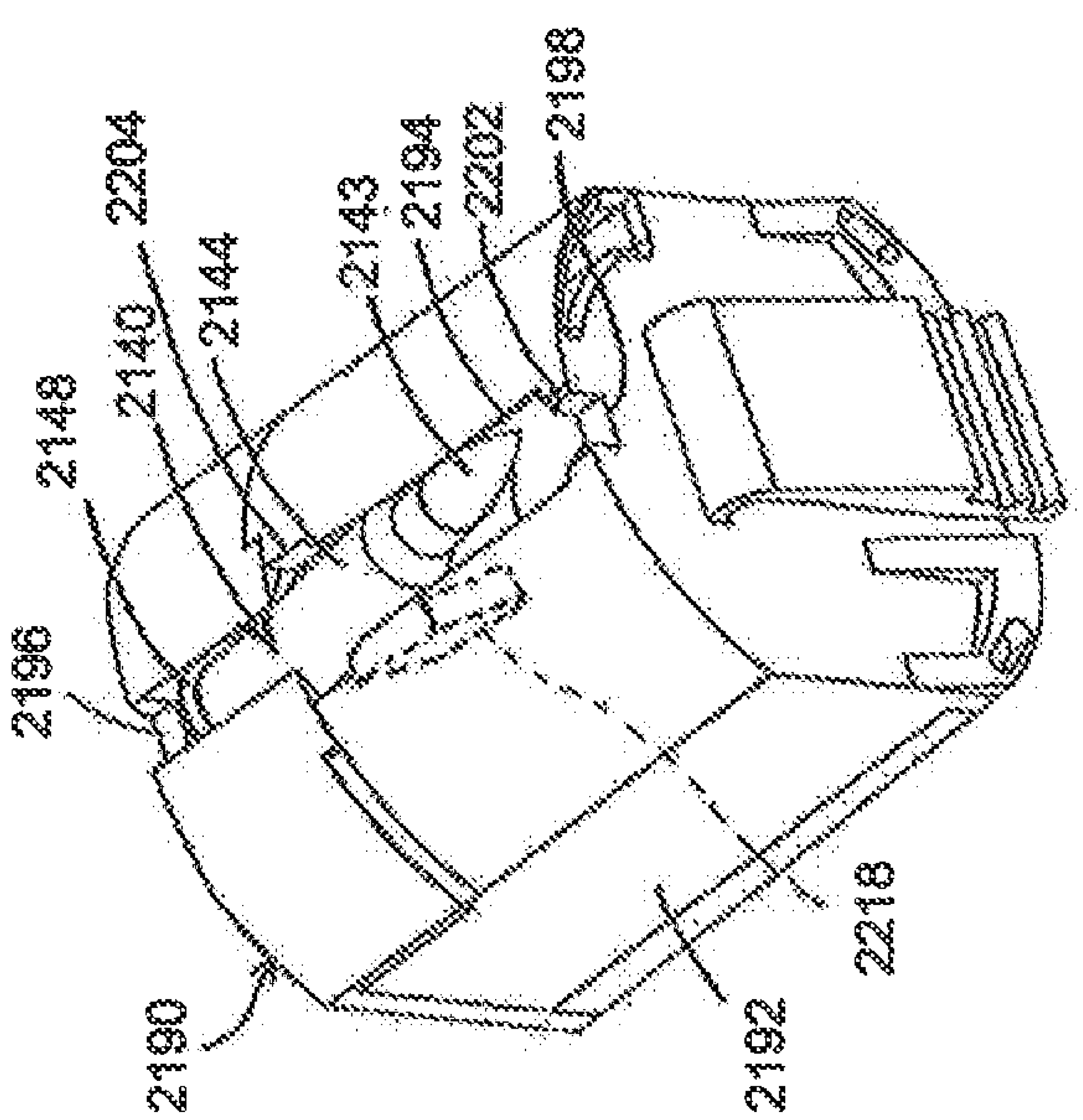

FIG. 88B

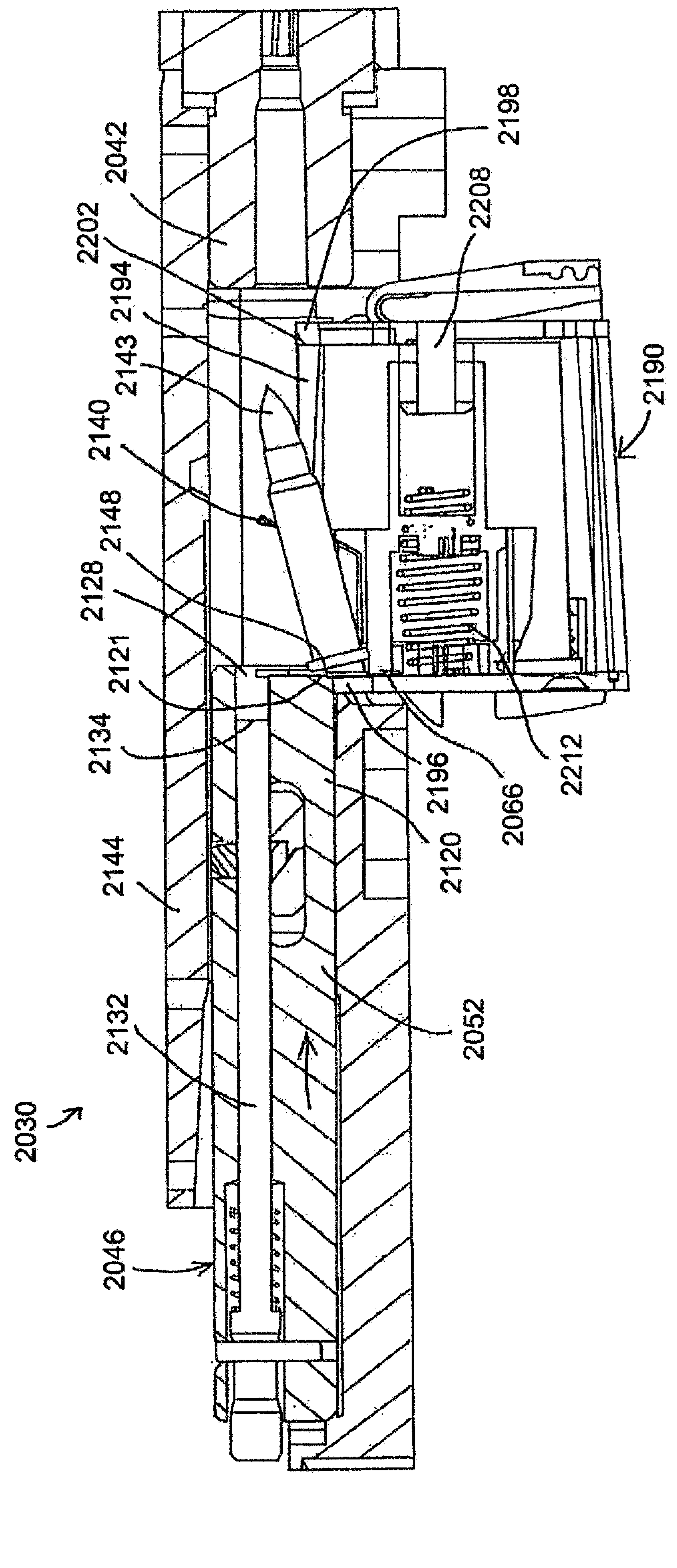

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FIG. 89A

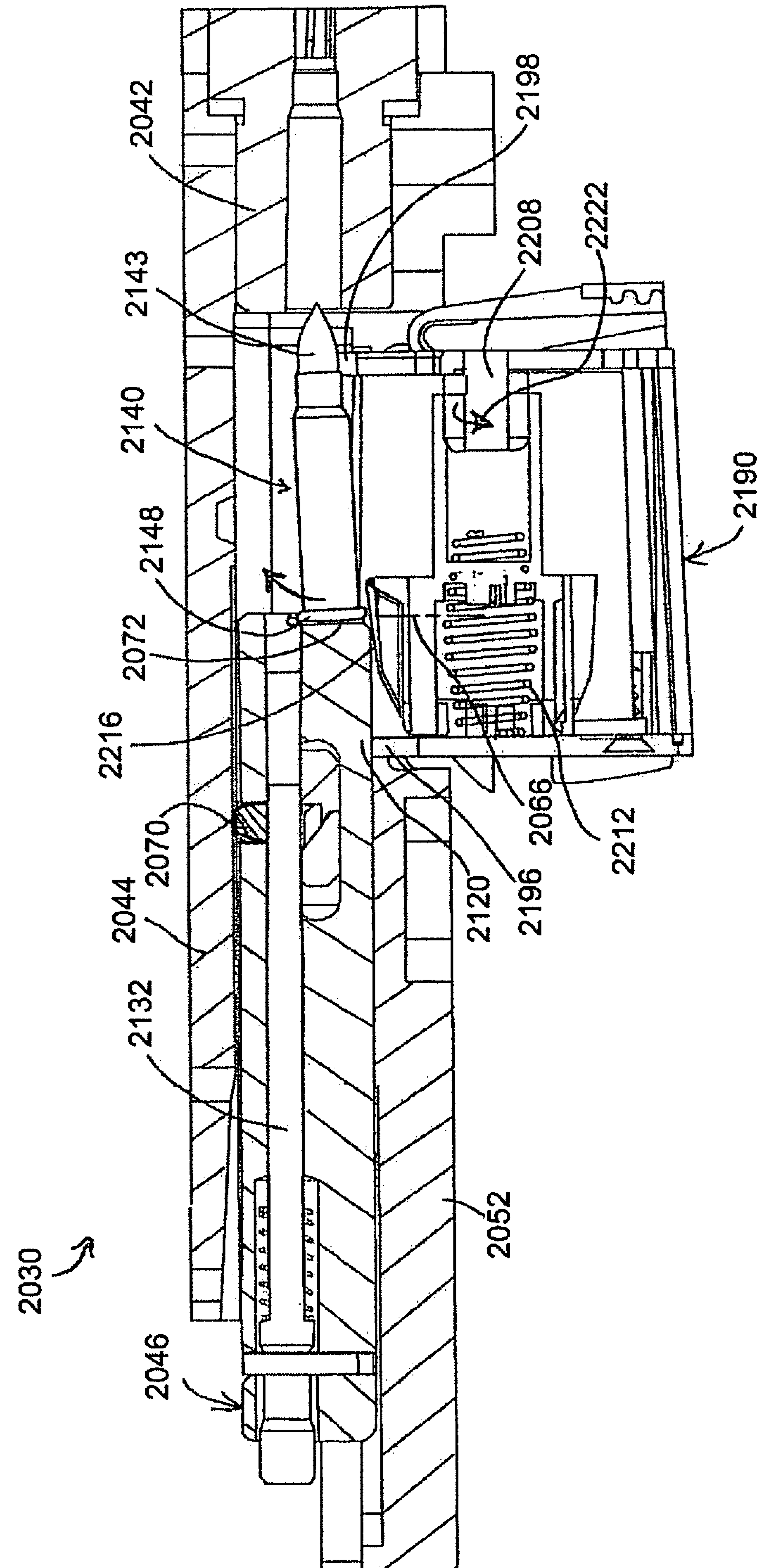
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FIG. 89B

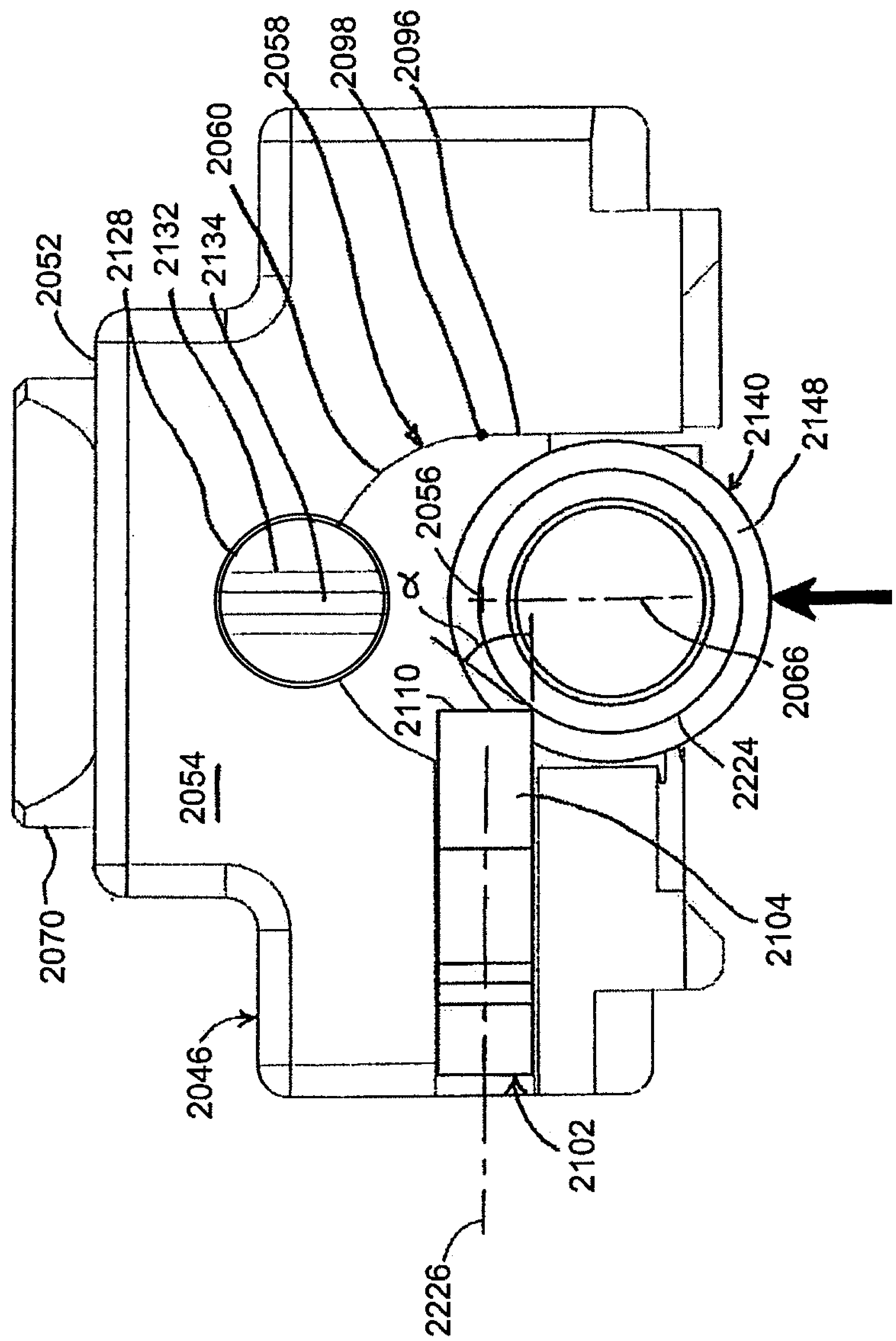

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FIG. 89C

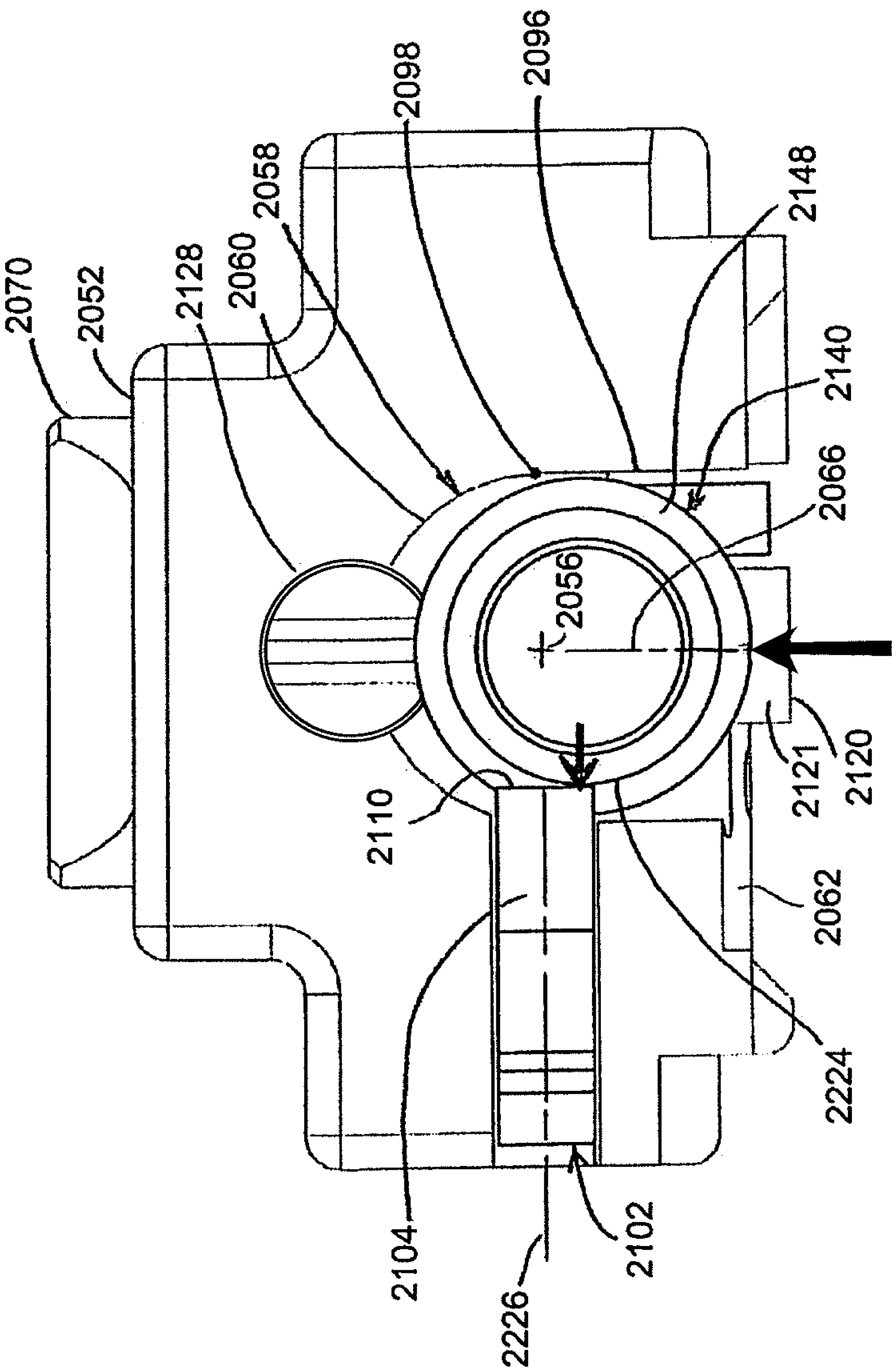
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FIG. 89D

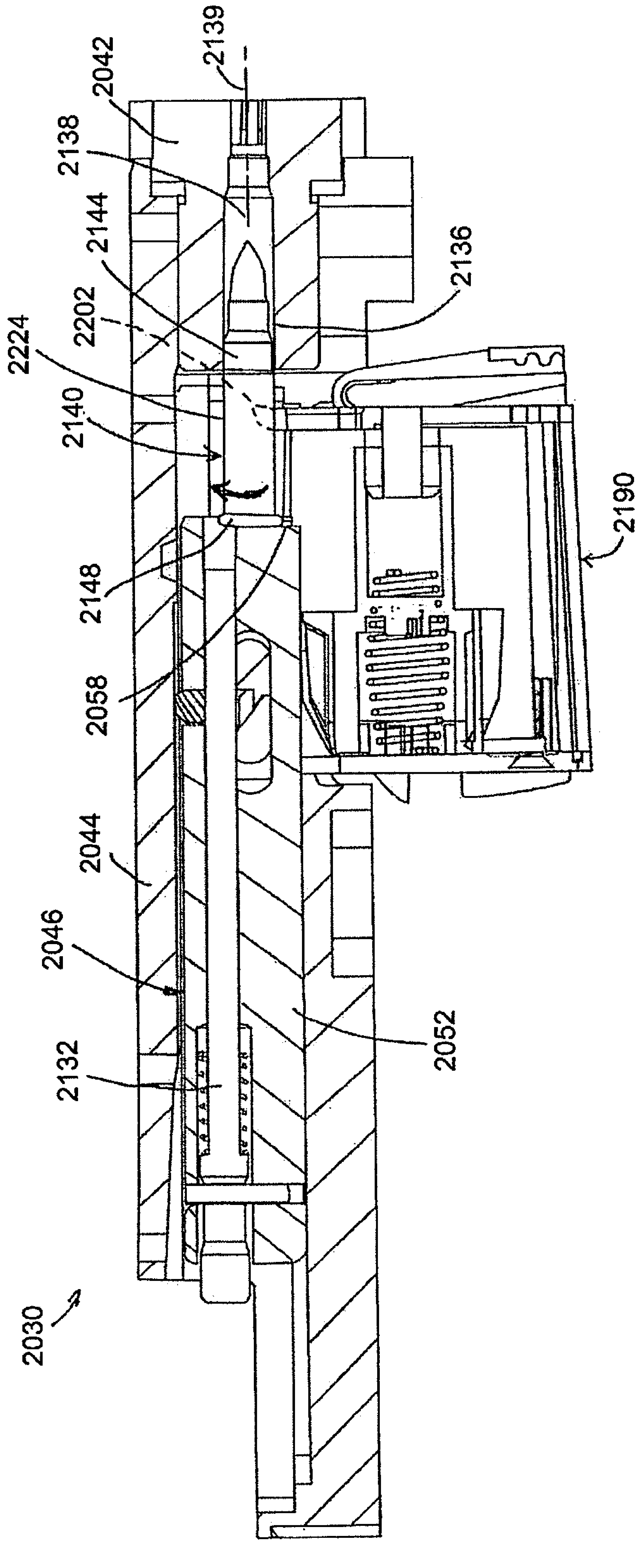
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FIG. 89E

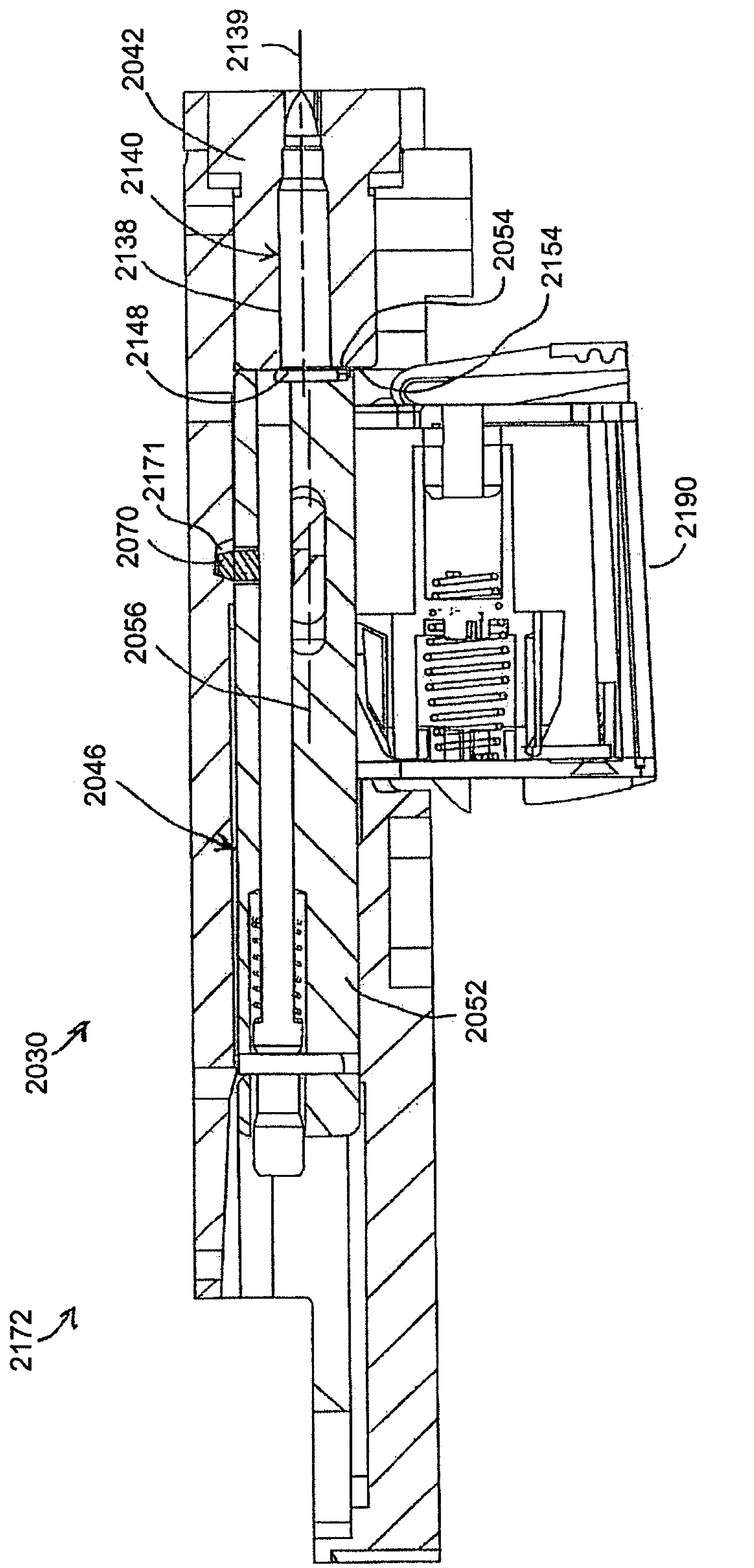

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FIG. 89F

SEMIAUTOMATIC FIREARM

RELATED APPLICATIONS

This application is a continuation of U.S. patent application Ser. No. 15/352,330, filed Nov. 15, 2015, now U.S. Pat. No. 10,788,277, which is a continuation of PCT/US2015/031210, filed on May 15, 2015, which claims priority to U.S. patent application Ser. No. 14/599,396, filed Jan. 16, 2015, Now U.S. Pat. No. 9,810,496, U.S. patent application Ser. No. 14/599,199, filed Jan. 16, 2015, now U.S. Pat. No. 9,599,417 and U.S. patent application Ser. No. 14/599,408, filed Jan. 16, 2015, now U.S. Pat. No. 9,513,076. This application also claims the benefit of U.S. Provisional Patent Application No. 61/993,541, filed May 15, 2014, U.S. Provisional Patent Application No. 61/993,563, filed May 15, 2014, and U.S. Provisional Patent Application No. 61/993,569, filed May 15, 2014. The above-recited applications are hereby incorporated by reference herein in their entireties.

BACKGROUND OF THE INVENTION

The inventions herein relates to semiautomatic firearms. Certain aspects relate to cycling mechanisms for such firearms. Such mechanisms require that the cartridge be retained in the firing chamber essentially until the bullet has left the barrel or the projectile velocity and performance will be impaired. These cartridges, for example the .17 Winchester Super Magnum (WSM) and the .17 Hornady Magnum Rimfire (HMR) are relatively inexpensive compared to high power centerfire cartridges and therefore have high consumer appeal for the recreation sport shooting market. Traditional rimfire semiautomatic recycling mechanisms generally rely on the weight of the bolt for providing a delay in "blowback" of the bolt. These mechanisms have not been proven suitable for high power necked rimfire cartridges due to the higher power and much greater rearward blowback force associated with these cartridges. Such mechanisms, for these cartridges, do not provide enough delay in the blowback of the bolt or the bolt weight is excessively heavy. The cycling mechanisms for the more powerful necked centerfire cartridges are not suitable either in that the rimfire cartridges generally do not provide sufficient gas pressures for such mechanisms, for example, gas operated cycling mechanisms used in AR-15 type rifles. Even if such mechanisms could be adapted to the necked rimfire cartridges, such mechanism are complicated, requiring many moving parts and thus would be relatively expensive; particularly compared to semiautomatic rifles for non-necked .22 caliber cartridges. Previous attempts at reasonably priced consumer oriented semiautomatic rifles for these high power rimfire cartridges have had performance issues, such as jamming and out-of-battery firing of cartridges. A reliable, mechanically simple, semiautomatic firearm with improved performance, particularly for high power necked rimfire cartridges, would be welcome. Moreover, enhanced safety and redundant systems are also welcomed.

Inventions herein relate to firearm extraction mechanisms, particularly for semi-automatic firearms. Such mechanisms often rely on a somewhat tenuous arrangement for securing a shell casing to a bolt of the firearm. The uncertainties associated with manufacturing tolerances of cartridges, as well as the spurious nature of the frictional forces exerted thereon, leads to instability during the extraction process that can cause failures to eject and sporadic ejection patterns. This can particularly be a problem when handling smaller diameter casings that are generally associated with rimfire cartridges (i.e., cartridges that are fired by impingement of a firing pin near the periphery of the base of the cartridge), particularly higher powered rimfire cartridges.

Also, the instability of traditional extractor mechanisms is more problematic when the retracting bolt speed is variable. Where the bolt is moved too slowly the cartridge case can become instable long before it's delivered to the ejector.

An ejector mechanism that overcomes these problems would be welcomed.

SUMMARY OF THE INVENTION

Improved mechanisms operate together to provide a highly reliable and robust semiautomatic firearm particularly suited to high powered rimfire cartridges. In particular embodiments, a system delays blowback in firearms with reciprocating bolt assemblies in semiautomatic firearms and is particularly suitable for high power necked rimfire cartridges. A feature and advantage of embodiments of the invention is that enhanced reliability and minimization of out-of-battery firing of cartridges is provided. In embodiments, a semiautomatic firearm utilizes cooperating and common components to provide both a delayed blowback and a lockout of the firing pin when a bolt assembly is out-of-battery.

In embodiments of the invention, a movable member within a bolt body functions as a blocking member that blocks the firing pin and preventing the firing pin from striking a cartridge when the bolt is not in battery. In embodiments, the firing pin has two stop portions that the movable member can engage depending on the cycle status of the firearm. One stop portion, when blocked, prevents the firing pin from traveling into cartridge headspace when the hammer receiving end of the firing pin is struck by the hammer, the other stop portion, when blocked, prevents the firing pin from retracting to the ready to fire position such that the hammer receiving end of the firing pin is not exposed and thus cannot be struck.

In embodiments, in an in-battery position, when the bolt assembly is closed on the firing chamber, a movable blocking member is in a non-blocking position with respect to the firing pin. The movable blocking member has a projecting portion extending from the bolt body to be removably received in a recess of the firearm housing, for example a ceiling of the receiver. The movable blocking member may be biased to urge the projecting portion outwardly into the recess. When the blocking member is in the non-blocking position the firing pin is free to travel past the blocking member and into headspace of the bolt body to achieve ignition. In embodiments, the blocking member may be engaged with a spring assembly for providing the bias. In embodiments, a separate blocking member carrier or cammed intermediary member, movably forwardly and rearwardly in the bolt body, may be engaged with a spring assembly to provide the outwardly bias to the blocking member through the intermediary member. The intermediary member may have a ramp portion that the blocking member rides up with the spring assembly urging the ramp portion against the blocking member thereby urging the blocking member to ride up the ramp.

In embodiments, the bias urging the blocking member outwardly may be manually removed by a manual handle, for example by manually retracting the ramp portion that is engaged with and urging the blocking member outwardly. The manual handle may be moved slightly rearwardly to back off the ramp and allow the movable member to retract to again put the blocking member in a blocking position with respect to the firing pin. Further motion of the manual handle then can pull the bolt assembly rearwardly to eject a cartridge engaged with the bold assembly.

In embodiments of the invention, a movable member, such as a lug, performs a locking function with respect to the in-battery position of the bolt assembly such that upon firing there is a delay in the retraction of the bolt assembly while the movable member unlocks. The movable member may extend from the bolt body outwardly to engage a recess in the firearm housing and be retractable inwardly between, respectively, a locked and an unlocked position. The movable member can extend and retract along an axis normal or transverse to the axis of the reciprocating bolt assembly and the axis as defined by the barrel bore. The movable member may have a bias towards the extended-locked position. In an embodiment the bias is provided by a ramp portion that is biased forwardly by a recoil spring assembly pushing a wedge under the movable member providing the bias outwardly. When the movable member is received in the recess in the housing an outwardly facing cam surface of the movable member engages a transition cam surface (an inclined surface) of the firearm housing requiring the transition cam surface to push the movable member inwardly with respect to the bolt assembly in order to escape the recess. Such inward movement requires retracting of the ramp portion within the bolt body, and due to the change of direction of the force, from normal or transverse to the axis of the reciprocating bolt assembly to a direction parallel to said axis, requires substantially more force than the force to overcome a force provided by the recoil spring assembly, for example, under the movable member. The movable member attempting to push the ramp portion downwardly is, appropriately termed, a reverse cam mechanism. The downward force increases the frictional resistance between the ramp portion and the surface upon which it slides and only a component of the downward force is translated to move the ramp portion. This component acts against and must overcome the frictional resistance as well as any additional spring force provided to resist the movement.

A feature and advantage of embodiments of the invention is that a cam mechanism is utilized in a normal forward fashion in association with a manual handle in a reciprocating bolt assembly of a semiautomatic firearm and is used in a reverse manner to delay blowback.

A feature and advantage of embodiments of the invention is a movable member slidingly constrained within the bolt body that locks out the firing pin, the movable member can be moved with respect to the lockout of the firing pin by cam surfaces engaging opposing ends of the movable member. A feature and advantage is that the movable member may be block shaped, a lug, with a firing pin opening therethrough that provides the firing pin block. A feature and advantage is that the block shaped movable member is not attached by pins or the like within the bolt and is simply slidingly constrained within open spaces in the bolt and breech region. Such a configuration eliminates wear issues, dirt and debris issue, and lubrication associated with using joints and pivot points for constraining moving parts. Moreover, in that the movable member has opposite ends which both are utilized as cam follower surfaces, this "float" of the movable member, with some free play in the constraint of the movable member, facilitates even engagement of the cam surfaces which the cam followers follow.

In embodiments, the movable member has a projecting locking tab that extends to engage a recess or stop surface in the ceiling of the receiver. The firing pin is blocked, or locked out, except when the tab is extending. The only recess or place for the locking tab to extend correlates to an in-battery position of the bolt assembly in an engaged ready-to-fire position.

A feature and advantage of embodiments of the invention is that mechanisms are utilized for delaying blowback that are contained mostly within the bolt body and therefore have minimal or reduced exposure to firing byproducts and contaminants.

A feature and advantage of embodiments of the invention is that the bias provided to the delayed blowback mechanism is a readily accessible spring that may be easily inspected and replaced if necessary.

In embodiments of the invention, the bolt assembly can be manually moved forwardly and rearwardly and pushed into an in-battery position if it is not in such a position. A manual handle extends from the bolt assembly and is directly engaged with the movable member carrier that traverses from the left side of the bolt body to the right side. The movable member carrier is moveable in a forward and rearward direction a limited amount within the bolt body, the limited movement associated with moving the movable blocking member up and down the ramp. The movable blocking member carrier may have the ramp surface, effectively a cam surface, engaged with the movable blocking member such that forward and/or rearward movement of the movable blocking member carrier moves the cam surface and a cam follower surface on the movable blocking member engaged with the cam surface causes the movable blocking member to move upwardly or downwardly.

A feature and advantage of embodiments of the invention is a semiautomatic firearm with a blocking member as part of a bolt assembly that moves in a vertical direction, up and down, between a blocking and non-blocking position with the firing pin. The blocking member needs to be in an upward or extended position, the non-blocking position, for the firing pin to be able to be struck by the hammer and translate forward to reach and impact a cartridge in a firing chamber.

A feature and advantage of embodiments of the invention is that particular components have multiple functions, thereby saving weight and minimizing the number of moving parts. For example, the movable member provides a delayed blowback mechanism for the bolt assembly and also provides a blocking or lockout for the firing pin when the bolt is not fully in the in-battery position. Moreover, the recoil spring assembly connected between the bolt assembly and the firearm frame may provide at least two functions. First the spring assembly provides forward bias to the bolt such that as the bolt is blown backwards the spring assembly cycles the bolt assembly forwardly, and second, the spring assembly provides an outward bias to the blocking member an intermediary member that engages the blocking member with a cam surface to urge the blocking member upwardly.

A feature and advantage of embodiments of the invention is that a delay in the bolt assembly blowback is affected by the disengagement of a projection extending from the bolt engaging the receiver that secures the receiver in an in-battery motion translation mechanism operating against spring aligned in the axis of the bolt assembly. A feature and advantage of embodiments is that the projection can be manually retracted from the receiver with a handle connecting to member that, by way of a ramp or cam surface extends and retracts the projection which allows opening of the bolt.

A feature and advantage of embodiments of the invention is that delay in the bolt assembly blowback is provided first by two serially connected motion translation mechanisms, in particular, two cam/cam follower mechanisms. In an embodiment, a spring bias is provided to the second cam follower mechanism. Moreover, the delayed blowback mechanism does not require a connection to the receiver by the mechanism, only a cam/cam follower engagement.

A feature and advantage of embodiments of the invention is the simplicity in that a single component both engages with a stop portion fixed with respect to the receiver for locking the bolt assembly in an in battery position and also engages with the firing pin to block the firing pin when the bolt assembly in an out of battery position. The movable member alternately locks the bolt and then blocks the firing pin providing simplicity of design and enhanced reliability, which is particularly effective in minimizing out-of-battery misfires.

In embodiments, a semiautomatic firearm with a reciprocating bolt in a receiver, the bolt may be manually retracted by pulling rearwardly a manual handle attached to a ramp portion that releases the engagement of the ramp with an outwardly projecting movable locking member with a stop portion fixed with respect to the receiver. The manual handle first releases the locking member with the rearward manual force on the handle, and then moves the bolt rearwardly with the continuing rearward manual force on the handle. Moreover, releasing the outwardly projecting movable locking member then blocks the firing pin decreasing out of battery misfires.

In embodiments of the invention, a bolt assembly of a semiautomatic firearm comprises a firing pin that has travel between a ready-to-fire position, a fire position, and two intermediate positions where it may be held or blocked. In embodiments the firing pin is held in the two intermediate positions by a blocking member that moves into and out of a blocking position. In embodiments the blocking member is the movable member that moves inwardly and outwardly with respect to the bolt body as the bolt assembly travels forwardly and rewardly.

In embodiments of the invention, a method of delaying the blowback of a bolt in an in-battery position after firing is provided by constraining a movable member within the bolt, the member movable in a direction transverse to the blowback direction of the bolt, positioning the movable member in a recess fixed with respect to a frame constraining the bolt, whereby the bolt cannot blowback until the movable member retracts, and providing a bias to resist the retraction, In embodiments the bias may be provided by a spring operating directly on the movable member. In embodiments, the method further may having the movable member engaging with a ramp portion such that the ramp portion must be moved, pushed out of the way, by the movable member pressing against a ramped surface of the ramp portion. In embodiments further comprising biasing the ramp portion to resist the movable member pushing it out of the way. The method comprising pushing the ramp portion out of the way to retract the movable member from the recess which then allows blowback of the bolt. Further, embodiments provide methods of locking out the firing pin depending on the positioning of the movable member.

Embodiments of the invention feature a sliding bolt assembly with a movable member with an outward projection, the outward projection having a sliding engagement surface. The movable member is part of the bolt assembly so it moves with the bolt assembly but also is movable in a direction transverse to the sliding direction of the bolt assembly (the bolt assembly moves forwardly and rearwardly). The movable member moves inward and outward by slidingly engaging a surface with structure on the receiver or firearm frame as the bolt reciprocates. In embodiments of the invention, the movable member provides an in-battery lock and provides a firing pin block when the bolt is not in the in-battery position. In embodiments of the invention, the outward projection is an end of two opposing ends, the opposite end of the outward projection, an inward sliding engagement surface that engages a wedge portion. The wedge portion is movable in the forward/rearward direction with respect to the bolt assembly and the wedge is biased forwardly to push the movable member outwardly. In embodiments the movable member floats within and is captured by a bolt body of the bolt assembly. In other embodiments the movable member is pivotal with respect to the bolt body.

A feature and advantage of the inventive aspects herein, such as the particular mechanisms and components accomplishing the delayed blowback and lockout of the firing pin, the extraction, and trigger pull adjustments, may be suitable for firearms that fire cartridges other than the necked rimfire cartridges such as .22 caliber rimfire (non-necked) and necked or non-necked centerfire cartridges.

Various embodiments of semiautomatic firearms with robust and redundant systems for reducing malfunctions are disclosed, particularly suitable for use withhigher powered rimfire cartridges, such as .17 HSR and .17 WSM. The embodiments disclosed herein may also be utilized in firearms that fire centerfire cartridges and in .22 caliber firearms. A safety trigger is provided that is passively actuated in advance of a firing trigger. The safety trigger maintains redundant safety mechanisms that prevent inadvertent or accidental actuation of the firing trigger. Accordingly, the firing trigger can be configured for actuation with a very low magnitude or “soft” pull without compromising safety. That is, conventional firearms require substantial pull to be actuated in order to assure that the trigger doesn’t misfire during otherwise routine handling. For the disclosed embodiments, the safety trigger assures that the firearm is discharged only upon deliberate actuation of the firing trigger. In one embodiment, a trigger pull adjustment mechanism provides adjustment of the pull of the firing trigger to a desired force required by the operator. The disclosed trigger pull adjustment mechanism reduces the number of components and complexity of the machined parts over conventional trigger pull adjustment mechanisms.

In some embodiments, a firearm with a safety trigger component must be retracted prior to the firing trigger being retracted to fire the firearm, the safety trigger providing a plurality of firing inhibitors. In one embodiment, the safety trigger component includes a direct hammer catch positioned in an interfering or catch position when the safety trigger is in an unretracted position and one or more additional firing inhibitors controlled by the safety trigger. In various embodiments, a firing inhibitor controlled by the safety trigger is a sear portion block. In some embodiments, the safety trigger moves a sear blocking portion between a blocking position and a non-blocking position with respect to the sear portion. Optionally, the sear portion is part of a unitary trigger component. In some embodiments, the safety trigger controls a firing trigger block that is positioned to prevent the pivoting of the firing trigger component about the pivot axis, thus inhibiting the retraction of the firing trigger.

Structurally, various embodiments of a trigger assembly of a firearm is disclosed, the trigger assembly including passive and redundant safety mechanisms to prevent unintentional firing when the firearm is in a firing mode. In some embodiments, the trigger comprises: a hammer rotatable about a first axis, the hammer including structure defining a capture feature; a firing trigger component rotatable about a second axis and including a first finger hook portion, the firing trigger component including a sear portion releasably coupled to the hammer; and a safety trigger component rotatable about the second axis and including a second finger hook portion, the second finger hook portion extending forwardly of the first finger hook portion. In some embodiments, a first of the redundant safety mechanisms includes a catch portion defined on the safety trigger component and, when the safety trigger is in a battery position, is aligned for arresting the capture feature of the hammer as the hammer rotates to prevent discharge of the firearm. In some embodiments, a second of the redundant safety mechanisms includes a blocking member operatively coupled with the safety trigger component for maintaining the blocking member in a blocking position when the safety trigger component is in a battery position, the blocking member blocking an underside of the firing trigger component when in the blocking position to prevent release of the sear portion from the hammer, the blocking member being operatively coupled with the safety trigger component for moving the blocking member out of the blocking position by moving the safety trigger out of the battery position to enable release of the sear portion from the hammer. In one embodiment, a rearward deflection of the safety trigger component causes rotation of the blocking member.

In certain embodiments, the blocking member includes an arcuate base portion rotatable about a third axis, the arcuate base portion defining a recess and being operatively coupled with the safety trigger component for rotation about the third axis. In one embodiment, the arcuate base portion blocks the underside of the firing trigger component from being actuated when the safety trigger component is in the battery position, and the recess aligns with the firing trigger when the safety trigger component is rotated out of the battery position to enable the firing trigger to release the hammer.

In some embodiments, the blocking member includes a lever portion operatively coupled with the safety trigger component for rotation about a third axis, wherein the lever portion blocks the underside of the firing trigger component to prevent disengagement of the firing trigger component from the hammer, the lever portion being maintained in the blocking position by the safety trigger when the safety trigger is in the battery position, the lever portion being selectively rotatable out of the blocking position by rotating the safety trigger out of the battery position.

Alternatively or in addition, the trigger assembly comprises a manual safety mechanism actuated by a push button forward of the first finger hook portion and laterally actuated for selectively placing the firearm in one of a safety mode and a firing mode, the manual safety mechanism being operatively coupled to the blocking member for preventing the safety trigger component from moving the blocking member out of the blocking position when in the safety mode, and enabling the safety trigger component to move the blocking member out of the blocking position when in the firing mode.

For embodiments including the fore-mentioned manual safety mechanism, the blocking member can include an arcuate base portion rotatable about a third axis, the arcuate base portion defining a recess and being operatively coupled with the safety trigger component for rotation about the third axis, wherein: the arcuate base portion blocks the underside of the firing trigger component from being actuated when the safety trigger component is in the battery position and when the firearm is in the safety mode and in the firing mode; and the recess aligns with the firing trigger when the firearm is in the firing mode and the safety trigger component is rotated out of the battery position to enable the firing trigger to release the hammer. Optionally, the lever portion that extends from the arcuate base portion of the blocking member.

In some embodiments, the blocking member includes a lever portion operatively coupled with the safety trigger component for rotation about a third axis, wherein the lever portion blocks the underside of the firing trigger component to prevent disengagement of the firing trigger component from the hammer, the lever portion being maintained in the blocking position by the safety trigger when the safety trigger is in the battery position and the firearm is in the firing mode, the lever portion being selectively rotatable out of the blocking position when the firearm is in the firing mode by rotating the safety trigger out of the battery position. In some embodiments, the lever portion contacts the firing trigger when the safety trigger is in the battery position.

In various embodiments, the firearm includes a bolt assembly translatable forwardly and rearwardly, the bolt assembly including a firing pin that is offset from the barrel axis for firing rimfire cartridges, and wherein the chamber is configured for necked cartridges. Some embodiments provide for arresting the hammer to facilitate semi-automatic operation. In various embodiments, a trigger pull adjustment mechanism is provided for adjusting a pull required to actuate the firing trigger component.

In various embodiments of the disclosure, a firearm having a fully cocked configuration and a triggered configuration is disclosed, comprising: a hammer including a sear engagement portion; a biasing element operatively coupled with the hammer that shifts the hammer from a first orientation that corresponds to the fully cocked configuration to a second orientation that corresponds to the triggered configuration; a firing trigger component including a sear portion that engages the sear engagement portion of the hammer when the trigger assembly is in the fully cocked configuration, the firing trigger component being actuatable for disengagement of the sear portion from the sear engagement portion, enabling the biasing element to shift the hammer from the first orientation to the second orientation; a safety trigger component selectively movable between a blocking position and a non-blocking position; and a blocking member that engages the safety trigger component and is moveable by the safety trigger component between a first position wherein the safety effector member prevents actuation of the firing trigger component when the safety trigger component is in the blocking position and a second position wherein the safety effector member enables actuation of the firing trigger component when the safety trigger component is in the non-blocking position.

The safety trigger component can optionally comprise a catch that prevents the hammer from reaching the second orientation from the first orientation when the safety trigger component is in the blocking position. The manual safety mechanism can include a safety bar accessible from outside the housing. In some embodiments, a housing contains the hammer and the biasing element, wherein the blocking member is selectively engageable with the housing to prevent the safety trigger component from moving the safety effector member. The blocking member can operatively coupled with a manual safety mechanism that selectively engages the safety effector member with the housing. The firing trigger component can be actuatable by rotation about a pivot, the pivot being operatively coupled with the housing.

In various embodiments of the disclosure, a semiautomatic firearm is presented having a fire trigger with a curvature and a central slot and a safety trigger disposed in the slot and having a curvature conforming to the curvature of the fire trigger, the fire trigger having a normal position and a fire position rearward of the normal position, the safety trigger having a normal position extending forwardly of the normal position of the fire trigger, and a fire position at or rearwardly of the normal position of the fire trigger, the safety trigger associated with at least two firing inhibitors, the firing inhibitors in a inhibiting position when the safety trigger is in the normal position and in a non-inhibiting position when the safety trigger is in the fire position.

Various embodiments of the disclosure include a hammer that pivots about a pivot axis and has capture features on opposing sides. In some embodiments, the hammer includes a first engagement portion that operates as a hammer to prevent the hammer release unless a safety trigger is retracted, and the hammer includes a second engagement portion as an arrestor that prevents automatic firing action and captures the hammer should the firing trigger remain retracted during a recoil cycle.

Some embodiments of the disclosure include a semiautomatic firearm suitable for high powered rimfire cartridges that incorporates a trigger assembly with a plurality of firing inhibitors to minimize misfires and out-of-breach firings of cartridges and that still allows for a low pressure trigger pull that can be adjusted by the user, for example, field adjustable.

Some embodiments disclose a semiautomatic firearm having a fire trigger with a curvature and a central slot and a safety trigger disposed in the slot and having a curvature approximating the curvature of the fire trigger, the safety trigger being connected to a plurality of firing inhibitors that each have an inhibiting position and a non-inhibiting position.

In various embodiments, a semiautomatic firearm is disclosed having a fire trigger with a curvature and a central slot and a safety trigger disposed in the slot and having a curvature substantially conforming to the curvature of the fire trigger, the fire trigger having a battery position and a fire position rearward of the battery position, the safety trigger also having a battery position extending forwardly of the battery position of the fire trigger, and a fire position at or rearwardly of the battery position of the fire trigger, the safety trigger associated with at least two fire inhibitors, the fire inhibitors being in an inhibiting position when the safety trigger is in the battery position and in a non-inhibiting position when the safety trigger is in the fire position.

Various embodiments of the disclosure provide a mechanism for stably securing a spent cartridge casing to a bolt assembly during extraction. In some embodiments, the same mechanism provides stability for a cartridge that is inserted onto a bolt assembly for the reloading process. In some embodiments, the extraction mechanism is tailored to accommodate high powered small caliber rounds, such as, for example, .17 Hornady Magnum Rimfire (.17 HMR) and .17 Winchester Super Magnums (.17 WSM) cartridges.

Various embodiments of the disclosure address the instability of traditional extractor mechanisms when the retracting bolt speed. Positive cartridge/casing retention of the extractor allows the system to not be speed dependent.

Structurally, an extraction mechanism for a firearm is disclosed, comprising a bolt assembly including a bolt with a bolt face, the bolt assembly being translatable along a central axis. A recess sized to accommodate the base of a cartridge is defined on the bolt face, the recess including a base surface on the bolt face, the base surface being substantially normal to the central axis. In various embodiments, the recess defines an access on a lateral face of the bolt. The access can be concentric about a lateral axis. A ledge portion partially surrounds the base surface of the bolt face, the ledge portion including an inclined face that defines a normal vector including an axial component parallel to the central axis that is directed toward the base surface. In various embodiments, the axial component is in the range of 40 degrees to 70 degrees inclusive from the normal vector. In one embodiment, a cross-section of the inclined face is substantially straight, so that the inclined face and the base surface define an acute angle therebetween. In one embodiment, the ledge portion includes an arcuate segment about the central axis, and can also include a substantially straight portion tangential to the arcuate segment. A retractable extractor can be disposed proximate the recess, the retractable extractor being extendable over the base surface. In one embodiment, a firing pin that selectively extends into the recess in a direction normal to the base surface.

The inclined face of the ridge enables the spent cartridge casing to be adequately secured to the bolt face, while enabling the spent cartridge casing to slide upward and outward from the recess when brought into contact with the ejector.

In various embodiments, a semi-automatic firearm is disclosed, comprising a barrel defining a chamber centered about a barrel axis for holding a rimfire cartridge and a bolt assembly operatively coupled to the barrel. The bolt assembly is movable along the barrel axis to an engagement position with the barrel and is adapted to discharge the rimfire cartridge. The bolt assembly can comprise a unitary bolt body having a distal end portion, the distal end portion defining a recess for receiving a head of a rimfire cartridge. The recess is bound by a base surface that is normal to the barrel axis, an undercut portion that extends distally from the recessed base surface, and a ledge portion distal to the undercut portion that protrudes radially inward toward the barrel axis relative to the undercut portion. The ledge portion defines a central axis and includes an inclined face that faces the base surface. In one embodiment, the inclined face presents a rearwardly facing partial frusto-conical surface for engaging an exposed portion of a rim of a rimfire cartridge in the recess. In various embodiments, the semiautomatic firearm is in combination with a rimfire cartridge.

In one embodiment, the recess sized for receiving a head of a rimfire cartridge. The base surface and undercut portion can be sized such that the head of the rimfire cartridge is slidable on the base surface in all radial directions from the central axis for positioning a rim of the rimfire cartridge to contact the inclined face of the ledge portion. In one embodiment, the barrel axis and the central axis are nonconcentric for seating a rim of a rimfire cartridge against the inclined face of the ledge portion when the bolt assembly is in the engagement position with the barrel. An extractor can be pivotally engaged with the bolt body, the extractor having a hook portion biased toward the central axis and extending over the recess. The hook portion can configured for engagement with a spent cartridge casing to push a rim of the spent cartridge casing into engagement with the inclined face of the ledge portion.

In various embodiments, at least part of the ledge portion is diametrically opposite the extractor. In one embodiment, at least a portion of the ledge portion is opposite the extractor, the hook portion being positioned for engaging a case wall of a rimfire cartridge to slide the rimfire cartridge on the base surface of the recess to contact with the inclined face of the ledge portion. In some embodiments, the ledge portion defines an arcuate segment.

The semi-automatic firearm can further include a receiver operatively coupled to the barrel, the bolt assembly being movably engaged within the receiver, the firearm including an ejector member positioned in an opening of the bolt body for ejecting a spent cartridge casing from the recess when the bolt assembly moves rearwardly.

In one embodiment, the inclined face of the ledge portion defines an acute angle facing inwardly toward the central axis. In various embodiments, the acute angle can be in a range of 25 degrees to 85 degrees inclusive, 25 to 65 degrees inclusive, 35 to 60 degrees inclusive, 35 to 55 degrees inclusive, or 40 to 50 degrees inclusive. In various embodiments, the ledge portion extends at least 30 degrees and less than 180 degrees around the recess.

In some embodiments, the recess is sized for a .22 caliber or smaller cartridge. The bolt recess can be sized to enable movement of the head of the cartridge at least 4% of the diametric distance of a standard cartridge size at the head of the cartridge.

In various embodiments of the disclosure, a semi-automatic firearm is disclosed for firing rimfire ammunition, the firearm comprising including a barrel defining a chamber for receiving and firing a rimfire cartridge, a receiver operatively coupled to the barrel, and a bolt assembly operatively coupled to the receiver and adapted for loading, firing, and ejecting a rimfire cartridge. The bolt assembly is translatable rearwardly along a central axis to a rearward position for withdrawal of a cartridge casing from the chamber and ejection of the casing, the bolt assembly being translatable from the rearward position forwardly for loading a rimfire cartridge from a magazine into the chamber. The bolt assembly can comprise a bolt body with a forward bolt face, and a recess defined on the forward bolt face for receiving the head of a rimfire cartridge, the recess being proximally bound by a base surface on the bolt face, the base surface being substantially normal to the central axis, the recess surface being oversized compared to a head of a rimfire cartridge. In various embodiments, the recess defines an access on a lateral face of the bolt. The access can be concentric about a lateral axis that intersects the central axis at a right angle.

The bolt assembly can further include a ledge portion that partially surrounds the base surface of the bolt face, the ledge portion including an inclined face that defines a normal vector including an axial component parallel to the central axis that is directed toward the base surface. The axial component is in a range of 40 degrees and 70 degrees inclusive relative to the normal vector. A retractable extractor, such as a claw-type extractor, can be disposed proximate the recess, the retractable extractor being extendable over the base surface. In some embodiments, a firing pin, such as a rim-type firing pin, selectively extends into the recess in a direction normal to the base surface, the firing pin parallel to and non-concentric with the central axis to effect rimfiring of a rimfire cartridge.

The ledge portion optionally includes an arcuate segment and a substantially straight portion tangential to the arcuate segment. In one embodiment, the retractable extractor is substantially centered at a location diametrically opposed to a junction point of the straight portion and the arcuate segment.

In various embodiments of the disclosure, an extraction mechanism for a firearm is disclosed comprising a bolt assembly including a bolt with a bolt face, the bolt assembly being translatable along a central axis. A recess is defined on the bolt face, the recess being proximally bounded by a base surface on the bolt face, the base surface being substantially normal to the central axis. A ledge portion can partially surround the base surface of the bolt face, the ledge portion including an inclined face that defines a normal vector including an axial component parallel to the central axis that is directed toward the base surface. In some embodiments, the axial component is in the range of 40 degrees and 85 degrees inclusive from the normal vector.

In some embodiments, a retractable extractor is disposed proximate the recess, the retractable extractor being extendable over the base surface. In one embodiments, a firing pin that selectively extends into the recess in a direction normal to the base surface. Optionally, the bolt assembly defines an off-axis bore that is parallel to and non-concentric with the central axis, the firing pin being disposed in the off-axis bore, wherein the firing pin is a rim-type firing pin.

The ledge portion can include an arcuate segment, the arcuate segment defining a radius about the central axis. The ledge portion optionally includes a substantially straight portion tangential to the arcuate segment, wherein the retractable extractor is substantially centered at a location diametrically opposed to a junction point of the straight portion and the arcuate segments.

In some embodiments, there is a firing chamber distal to the bolt assembly, the firing chamber being concentric about a barrel axis. Optionally, the firing chamber includes structure defining a circular access opening and a ridge, the ridge including an edge that is immediately adjacent the circular access opening, wherein the retractable extractor engages the ridge to rotate the retractable extractor away from the recess when the firearm is in a firing position. The central axis and the barrel axis can be parallel and non-concentric. In one embodiment, the central axis and the barrel axis are spaced apart and the ledge portion is dimensioned for engagement of a cartridge rim with the inclined face of the ledge portion when the firearm is in a firing configuration.

In various embodiments of the disclosure, a firearm is disclosed, comprising a firing chamber distal to the bolt assembly, the firing chamber being concentric about a barrel axis; a bolt assembly including a bolt with a bolt face, the bolt assembly being translatable along a central axis, the central axis and the barrel axis being substantially parallel and non-concentric; a recess defined on the bolt face, the recess being proximally bounded by a base surface on the bolt face, the base surface being substantially normal to the central axis; and a ledge portion and an undercut portion that partially surrounds the base surface of the bolt face, the ledge portion extending towards the central axis relative to the undercut portion and defining an inclined face that faces the base surface, the ledge portion including an arcuate segment, the arcuate segment defining a radius centered about the central axis. The ledge portion is dimensioned and the central axis and the barrel axis are spaced apart for engagement of a cartridge rim with the inclined face of the ledge portion when the bolt is engaged with the firing chamber in a firing configuration. The inclined face of the ledge portion can define a frusto-conical headspace. Optionally, the inclined face can define a profile that is arcuate and convex.

In various embodiments, a method for extracting a spent cartridge casing from a firing chamber of a firearm includes providing a bolt translatable along a central axis and including a bolt face, a recess defined on the bolt face that includes a base surface that is substantially normal to the central axis, a ledge portion that partially surrounds the base surface, the ledge portion including an inclined face that defines a normal vector including an axial component parallel to the central axis that is directed toward the base surface, and a retractable extractor disposed proximate the recess that is extendable over the base surface; and

causing the retractable extractor to extend over the base surface as the bolt is translated away from the firing chamber for engagement with the spent cartridge casing, the engagement of the shell casing causing a rim portion of the spent cartridge casing to engage the inclined face of the ledge portion, thereby capturing the rim portion of the shell casing.

The various mechanisms provide for a highly robust, reliable, semiautomatic firearm.

DESCRIPTION OF THE DRAWINGS

FIG. **1**A is a perspective view of a semiautomatic firearm in accord with the inventions herein.

FIG. **1**B is a more detailed perspective view of the semiautomatic firearm of FIG. **1**.

FIG. **2** is a perspective view of a molded stock for receiving the receiver, barrel, and trigger and firing mechanism of the firearm of FIG. **1**.

FIG. **3**A is a perspective view of the firearm of FIGS. **1** and **2** with the stock and portions removed.

FIG. **3**B is a side elevation view of the firearm of FIG. **3**A with portions including the receiver removed.

FIG. **4** is an exploded view of components of a firearm in accord with the inventions herein.

FIG. **5** is a cross sectional view of the firearm of FIG. **3**A taken through the manual handle with the outwardly projectable movable member in a recess in the ceiling of the receiver in a non-blocking position with respect to the firing pin.

FIG. **6** is a cross-sectional view of the receiver of FIGS. **4** and **5** illustrating a recess including a cam surface in the ceiling of the receiver for receiving/engaging the movable member.

FIG. **7** is a perspective view of a bolt assembly with the manual handle separated therefrom.

FIG. **8** is a perspective view of a bolt body taken from the right front corner FIG. **9** is a perspective view of the bolt body of FIG. **13** taken from the right rear corner.

FIG. **10** is a front elevation view of the bolt body of FIGS. **13** and **14**.

FIG. **11** is a cross-sectional view of the bolt body of FIG. **13** illustrating a spring recess for the bolt recycling spring assembly.

FIG. **12** is a cross-sectional view of a bolt body illustrating the apertures for the spanning member and the movable member.

FIG. **13** is an exploded perspective view of the components of the bolt assembly except for the bolt body for purposes of illustration.

FIG. **14** is an exploded perspective view of the manual handle and connection means to the bolt body.

FIG. **15** is a perspective view of the bolt assembly with the recoil spring assembly engaged therewith and with a necked rimfire cartridge in the bolt headspace.

FIG. **16** is a perspective view of the bolt assembly of FIG. **15** with the bolt body removed.

FIG. **17** is a cross-sectional through the bolt body and firing pin.

FIG. **18** is a side schematic elevation view of the semiautomatic firearm with a bolt locking mechanism.

FIG. **19** is a side schematic elevation view of the semiautomatic firearm of the FIG. **18** with the movable member unlocked.

FIG. **20** is a side schematic elevation view of the semiautomatic firearm with a bolt locking mechanism that has two reverse cam mechanisms.

FIG. **21** is a cross sectional view showing the bolt assembly mechanisms of FIG. **20** in detail.

FIG. **22**A is top schematic plan view of a semiautomatic firearm with the bolt assembly in an in battery position.

FIG. **22**B is a side schematic elevation view of the semiautomatic firearm of the FIG. **22**A illustrating the position of the movable member.

FIG. **23**A is top schematic plan view of the semiautomatic firearm of FIGS. **22**A and **22**B showing the position of the bolt assembly out of battery, for example after firing a cartridge.

FIG. **23**B is a side schematic elevation view of the semiautomatic firearm of the FIG. **23**A illustrating the position of the movable member.

FIG. **24**A is top schematic plan view of the semiautomatic firearm of FIG. **23**A with the bolt assembly in a full retracted position.

FIG. **24**B is side schematic elevation view of the semiautomatic firearm of the FIG. **24**A illustrating the position of the movable member.

FIG. **25**A is top schematic plan view of a semiautomatic firearm of FIGS. **22**A to **24**B after the bolt assembly has recoiled to the in-battery position and the locking member is received in the recess in the receiver.

FIG. **25**B is side schematic elevation view of the semiautomatic firearm of the FIG. **25**A illustrating the position of the movable member.

FIG. **26** is a top plan view of a movable/blocking member.

FIG. **27** is a perspective view of a movable/blocking member.

FIG. **28** is another perspective view of the movable/blocking member of FIGS. **26** and **27**.

FIG. **29** is a perspective view a carrier for the movable/blocking member of FIGS. **26-28**.

FIG. **30** is another perspective view of the carrier of FIG. **29**.

FIG. **31** is a perspective view of the movable member engaged with the ramp portion and in a non-blocking position with respect to the firing pin when the bolt assembly is in the in-battery position.

FIG. **32** is a perspective view of the assembly of FIG. **31** with the firing pin in the most forwardly position for firing the cartridge.

FIG. **33** is a perspective view of the assembly of FIGS. **31** through **32** with the outwardly projectable movable member lowered to be in a blocking position with the firing pin immediately after a cartridge is fired, the firing stop portion still forward of the movable member.

FIG. **34** is a perspective view of the assembly of FIG. **33** with the stop portion of the firing pin engaged with the movable member.

FIG. **35** is a perspective view of the assembly of FIGS. **31** through **34** with the movable member moving up the ramp from the position in FIG. **34** and the firing pin moving to a release position with the movable member.

FIG. **36** is a perspective view of the assembly of FIGS. **31** through **35** the movable member in the non-blocking position with respect to the firing pin.

FIG. **37**A is schematic plan view showing the position of the firing pin with respect to the bolt assembly in the in-battery position.

FIG. **37**B is schematic elevation view showing the position of the firing pin and movable member with respect to the bolt position of FIG. **37**A.

FIG. **38**A is schematic plan view showing the position of the firing pin with respect to the bolt assembly after ignition of a cartridge and out of the in-battery position.

FIG. **38**B is schematic elevation view showing the position of the firing pin and movable member with respect to the bolt position of FIG. **38**A.

FIG. **39**A is schematic plan view showing the position of the firing pin with respect to the bolt assembly in the full recoil position of the bolt assembly.

FIG. **39**B is schematic elevation view showing the position of the firing pin and movable member with respect to the bolt position of FIG. **39**A.

FIG. **40**A is schematic plan view showing the position of the firing pin with respect to the bolt assembly having returned to the in-battery position from the full recoil position with the firing pin rearward end exposed.

FIG. **40**B is schematic elevation view showing the position of the firing pin and movable member with respect to the bolt position of FIG. **40**A.

FIG. **41**A is schematic plan view showing the position of the firing pin with respect to the bolt assembly with the manual handle and carriage being moved rearwardly with respect to the bolt body.

FIG. **41**B is a schematic elevation view of the firing pin, movable member, and carrier, with respect to the position of FIG. **41**A.

FIG. **42** is a perspective view of the assembly of FIGS. **31** through **36** with the ramp portion manually moved rearwardly lowering the movable blocking member to a blocking position also as portrayed in FIGS. **41**A and **41**B.

FIG. **43** is a perspective view of the assembly of FIG. **42** where the hammer has struck the firing pin and the blocking portion precludes the firing pin from traveling forward to the headspace of the bolt thereby precluding firing of a cartridge in the headspace.

FIG. **44** is a side elevational view of an embodiment where the movable blocking member is attached to the bolt body at a pivot point.

FIG. **45** is a side elevational view of a firearm in an embodiment of the disclosure.

FIG. **46** is an exploded view of the firearm of FIG. **45**.

FIG. **47** is an exploded view of receiver and barrel of the firearm of FIG. **45**.

FIG. **48** is a detail view of the trigger assembly, bolt assembly, chamber, and barrel of a firearm with the receiver removed in an embodiment of the disclosure.

FIG. **49**A is an exploded view of the trigger assembly of FIG. **47** with trigger component cluster depicted as removed from a trigger mechanism housing.

FIG. **49**B is a top perspective view illustrating the interior of the trigger mechanism housing of FIG. **49**A.

FIG. **50** is an elevational view of a firearm with the stock and trigger assembly housing removed in an embodiment of the disclosure.

FIG. **51** is an exploded view of principal components of the trigger assembly in an embodiment of the disclosure.

FIG. **52** is a rear cutaway perspective view of the stock and trigger assembly of FIG. **50** with portions of the stock and trigger mechanism housing removed for illustration.

FIG. **53** is a forward looking right side perspective view of the principal components of the trigger assembly of FIG. **50** in isolation.

FIG. **54** is a rearwardly looking left side perspective view of the principal components of the trigger assembly of FIG. **50** in isolation.

FIG. **55** is a upwardly looking perspective view of the hammer assembly in isolation with the hammer spring extended.

FIG. **56** is a perspective view of a hammer, a shaft, a bushing, and a rotational spring in assembly in an embodiment of the disclosure.

FIG. **57** is a side elevation schematic view of trigger assembly components in a battery position, illustrating a cocked configuration of a firing sequence, where a firing trigger and a safety trigger are in a battery position in an embodiment of the disclosure.

FIG. **58** is the trigger assembly components of FIG. **57** in an enabled configuration of a firing sequence, where the firing trigger is in a battery position and the safety trigger rotated out of the battery position in an embodiment of the disclosure.

FIG. **59** is the trigger assembly components of FIG. **57** in a fired configuration of a firing sequence, where the safety trigger and the firing trigger are in a firing position in an embodiment of the disclosure.

FIG. **60** is the trigger assembly components of FIG. **57** where a firing trigger and a safety trigger are in a battery position and the safety trigger catches the hammer to prevent firing in an embodiment of the disclosure.

FIGS. **61**-**63** are a side elevation schematic views of the trigger assembly components and the operation of a blocking member during the firing sequence of FIGS. **57**-**59** in an embodiment of the disclosure.

FIGS. **64**-**66** are side elevational schematic views of the trigger assembly components during a cocking sequence to restore the trigger assembly from the triggered configuration to the fully cocked configuration in an embodiment of the disclosure.

FIG. **67** is a reverse front perspective view of the trigger assembly components and illustrating the arresting mechanism that facilitates semi-automatic operation in an embodiment of the disclosure.

FIG. **68** is a side elevational view of the trigger assembly components and arresting mechanism of FIG. **67**.

FIG. **69** is a side reverse rear perspective view of the trigger assembly components and arresting mechanism of FIG. **67**.

FIG. **70** is a schematic elevational view of operation of the arresting mechanism where the triggers become or remain actuated during the cocking of the firearm.

FIGS. **71**-**75** are side elevational schematic views of the trigger assembly components during the cocking sequence of FIGS. **64**-**66**, illustrating operation of the arresting mechanism in an embodiment of the disclosure.

FIG. **76** is a partially exploded cutaway view of a trigger pull adjustment mechanism in an embodiment of the disclosure.

FIG. **77** is an enlarged perspective view of a firing trigger return spring for the trigger pull adjustment mechanism of FIG. **76** in an embodiment of the disclosure.

FIG. **78** is a perspective view of an adjustment tool for use with the trigger pull adjustment mechanism of FIG. **76** in an embodiment of the disclosure.

FIG. **79** is a sectional view of the trigger pull adjustment mechanism of FIG. **76** in assembly and operation of the adjustment tool of FIG. **78** in an embodiment of the disclosure.

FIGS. **80** and **81** are sectional views of a conventional rotating claw extractor in operation;

FIG. **82** is a side view of a firearm utilizing an extraction mechanism in an embodiment of the disclosure;

FIG. **82**A is an enlarged partial view of the firearm of FIG. **82**;

FIG. **83** is a bolt assembly in an embodiment of the disclosure;

FIG. **84** is an elevation view of a distal end of the bolt assembly of FIG. **83**;

FIG. **85** is a sectional view of the bolt assembly of FIG. **84**;

FIG. **85**A is an enlarged, partial sectional view of the bolt assembly of FIG. **85**;

FIG. **85**B is an enlarged, partial sectional view of the bolt assembly in an alternative embodiment of the disclosure;

FIG. **85**C is an elevation view of a cartridge, including dimensions for a .17 WSM cartridge;

FIGS. **86**A and **86**B are plan sectional and elevation sectional views, respectively, of the bolt assembly, breech, and firing chamber in a firing position in an embodiment of the disclosure

FIG. **86**C is a front view of the bolt assembly with a cartridge in the firing position in an embodiment of the disclosure;

FIGS. **86**D through **86**F are sectional views of the bolt assembly, breech, and firing chamber for an extraction utilizing a blowback force at various stages of extraction in an embodiment of the disclosure;

FIG. **86**G is a front view of the bolt assembly with a spent cartridge casing secured thereto and corresponding to FIGS. **86**F and **87**B in an embodiment of the disclosure;

FIG. **86**H is a sectional view of the bolt assembly, breech, and firing chamber during ejection of the spent cartridge casing in an embodiment of the disclosure;

FIGS. **87**A and **87**B are sectional views of the bolt assembly, breech, and firing chamber for an extraction that could be associated with a non-blowback extraction at various stages of extraction in an embodiment of the disclosure;

FIG. **88**A is perspective views of a magazine for use in embodiments of the disclosure;

FIG. **88**B is a perspective view of the magazine of FIG. **88**A with a cartridge extending therefrom;

FIGS. **89**A, **89**B, **89**E, and **89**F are elevation sectional views of the firearm during a reloading sequence in an embodiment of the disclosure; and

FIGS. **89**C and **89**D are front elevation views of the bolt assembly and the cartridge during the reloading sequence of the firearm in an embodiment of the disclosure.

DETAILED DESCRIPTION

Referring to FIGS. **1**A-**4**, a semiautomatic firearm **30** according to embodiments of the invention is illustrated and generally comprises a housing **32** including a receiver **34**, a barrel **36** with a bore **37** and a firing chamber **38**, a stock **40** with a forestock portion **42**, an ejection port **44**, a trigger and firing assembly **46** with a hammer **47**, a bolt assembly **48**, a recoil spring assembly **50**, and a magazine **52**. In one example, the trigger and firing assembly may be inserted into the unitary stock and forestock component as shown in FIG. **2**. Then the barrel and upper receiver assembled on top of that and coupled to the trigger and firing assembly. The bolt assembly and recoil spring assembly inserted into the rear upward opening **56** of the receiver with panels added.

The bolt assembly **48** is slidingly engaged in the receiver **34** to move forwardly and backwardly along a bolt assembly travel axis aa which also is also coincident with a barrel axis ab of the bore **37** and is generally a central axis ac of the firearm. The receiver generally has an interior **57** defining a breech region that receives the bolt assembly, an opening **58** that defines the ejection port **44**, an inner surface **60** and a ceiling **61**. Ledges **62** on the receiver **34** constrain the bolt assembly and may provide bearing surfaces for sliding engagement with the bolt assembly. An engagement surface defining a longitudinal cam surface **63** is fixed with respect to the receiver and may be on the ceiling **61** of the receiver. The cam surface includes a first surface **64** that is at a first elevation **64**, a second displaced surface at a second elevation **65** that is configured as a recess surface **65**, and a transition cam surface **67** which provides an inclined surface leading from the first surface to the second surface. The first surface is part of a linear portion **69** as illustrated is an inward cam portion **64** with respect to the bolt assembly. The recess surface **65** defining an outward cam portion **65** and the transition cam surface **67** being an inclined surface. In other embodiments the cam portions and cam surfaces may be part of a rib extending inwardly in the breech area or a separate piece attached to the receiver.

Referring generally to FIGS. **1**-**17**, details of an embodiment of the bolt assembly **48** are illustrated, particularly showing features of the assembled bolt assembly. The bolt assembly has a forward face **68**, a top side **70**, a left side **72**, a right side **74**, a bottom side **76**, and a rearward side **78**. The bolt assembly comprises a bolt body **82** that may be a unitary form, a firing pin assembly **86**, a retractable extractor **88**, a manual handle assembly **90** with a manual handle **92**, a bolt locking mechanism **93** including a movable member **94** that moves upwardly and downwardly about an axis b transverse to the axis aa, which may be perpendicular to the axis b, and that engages the cam surface **63** of the inside surface of the receiver. In embodiments, the bolt assembly has a firing pin blocking mechanism **95**, discussed in detail below, which may utilize componentry of the bolt locking mechanism. The movable member has a cam follower surface **96** that engages a cam surface on the ceiling of the receiver. When engaged in the recess **66**, the bolt assembly is in a locked position with respect to the in-battery position. Unlocking the bolt assembly, requires disengagement of the cam follower surface with the recess. When out of the recess the bolt is in an unlocked position. The cam surface may be part of the receiver or a separate component attached to the receiver.

The bolt assembly **48** according to embodiments of the inventions, including the internal components, is illustrated in further detail in FIGS. **3**-**5**, **7**-**17**. The bolt body **82** has a firing pin opening or conduit **100** extending longitudinally through the bolt body that receives the firing pin assembly **86**. The firing pin thus moves longitudinally in the opening along an axis of that is generally parallel to the central axis, the axis of the bolt assembly, The conduit is defined by the internal surface **102** and includes a spring stop surface **104** where greater bore **106** transitions to a lesser bore **108**, see in particular FIG. **11**. A cartridge head space **112** is defined on the forward face **68** of the body (and bolt assembly) and is defined by lip **114** which extends over an undercut region **115** and is generally of an inverted U-shape, defining a cartridge head receiving region **118** with a flat surface that engages the cartridge **119**. As best illustrated in FIGS. **15** and **17**, the cartridge **119**, such as a necked rimfire cartridge, is received in the U-shaped recess and seats against the planar bolt head space surface, and is pushed into the undercut region by the retractable extractor **88**. This is further described in a related application. The cartridge **119** is a high power rimfire cartridge and has a casing **121** with a casing head **122** and a rim **123**. On the bullet end of the casing, a collar **125** and necked down portion **126** reduce the diameter of the casing to be sized for the bullet **127**. When used herein, "necked rimfire cartridge" refer to these cartridges. Such cartridges have the primer propellant in the rim and do not have a central primer. The barrel and firing chamber are configured for receiving the necked rimfire cartridge as illustrated in FIGS. **18** and **19**. The .17 HMR and .17 WSM are such cartridges.

The extractor and the cartridge head receiving region with the undercut has been found to reliably extract and eject cartridges in synergistic association with the componentry described herein and as such contribute to and are an integral part of providing a reliable, mechanically simple, semiautomatic firearm with improved performance, particularly for high power necked rimfire cartridges.

The trigger and firing mechanism **46** includes a double safety trigger **128** and pull adjustment **129**. These are described in detail in a related application. The double safety trigger and trigger pull adjustment have been found to contribute to and are an integral part of providing a reliable, mechanically simple, semiautomatic firearm with improved performance, particularly for high power necked rimfire cartridges.

An ejector slot **120**, shown best in FIGS. **8-10**, receive the ejector **124**, see FIG. **4**, which extends along the bottom side **76** of the body. The ejector is fixed with respect to the receiver and kicks out a spent casing that is held by the bolt assembly, when the bolt assembly is blown back, see FIG. **9**. Bearing surfaces **130**, **132** of the bolt body, as seen in FIG. **8**, engage the ledges **62** of the housing/receiver **34**, see FIGS. **4** and **5**. A pin aperture **136** for retaining the firing pin extends vertically through a rearward portion **137** of the bolt body, referencing FIGS. **8-12**. A pin aperture **138** also extends vertically through a forward portion **140** of the bolt body **82** for retaining the retractable extractor **88**. A slot **144** for receiving components of the extractor assembly extends horizontally inwardly on the right side of the forward portion of the bolt body. A further slot **148** for receiving components of the bolt locking mechanism and firing pin blocking mechanism (discussed below) extends through the forward portion of the bolt body from the left side to the right side and a slot on the top surface **150** of the bolt body guides and constrains the movable member **94**, which in embodiments is part of the bolt locking mechanism **93** and the firing pin block **95**. The movable member may thus be termed a movable blocking member or a blocking member or a locking member depending on context. The movable member may be said to "float" within the bolt body **82** in that it is only constrained and not fastened or directly attached to the bolt body. A longitudinally extending recoil spring assembly opening **151** extends from the rearward end to the slot **148** for the bolt locking mechanism and firing bin block.

The bolt assembly further has the manual handle **92** that extends out the ejection port **44** of the firearm. The manual handle is attached to an intermediary member **154** that has a side aperture **155** that is in alignment with the recoil spring assembly opening **151**. A carrier or spanning member **158** for the movable member **94** is inserted into the slot **148** and extends from the left side of the bolt body to the right side and engages the movable member within the bolt body. The carrier member has a side aperture **157** in alignment with the intermediary member aperture **155** as well as the recoil spring assembly opening **151**. The spanning member **158** has a ramp portion **159** with a ramp surface **161** that cooperates with a cooperating surface **160** on the movable member **94** such that as the ramp is moved forwardly or backwardly, the movable member raises or lowers respectively. The ramp surface acts as a cam surface and the movable member is a cam follower.

The firing pin assembly **86** and how it integrates with the bolt body is best seen in FIGS. **5**, **11**, **13**, **16**, and **17**. The firing pin assembly, as illustrated, includes an elongate shaft defining the firing pin **162** and has a forward cartridge engagement tip **164** that has a flattened elongate shape for engaging the rims of rimfire cartridges and a blunt rearward end **168** that is struck by the hammer **47** (see FIG. **3**). The firing pin has a pair of reduced diameter, or thinned, portions **172**, **174** that define a forward stop portion **176** and first forward stop surface **178**. Additionally, a second rearward stop portion **180** and respective second stop surface **182** is defined by the rearward reduced diameter or thinned portion **174**. A third intermediate stop surface **186** is positioned between the forward and rearward stop surfaces. The functionality of these are discussed below. The firing pin is retained in the opening **100** by way of a pin **188** secured in the pin aperture and extending through a slot **190** in the rearward end portion **192** of the firing pin. A spring **193** is positioned in the firing pin opening **100** between a spring stop **194** on the firing pin and the spring stop surface **104** defined in the bolt body. The spring **193** provides a rearward bias to the firing pin.

The recoil spring assembly **50** and how is integrates with the bolt assembly is best illustrated in FIGS. **3**, **5**, **7**, **13-16**. The recoil spring assembly has a shaft **204** that, in an embodiment as illustrated, is telescoping with an inner shaft portion **206** and an outer shaft portion **208**. A spring stop **209** is positioned on a forward end **210** of the shaft. A housing engagement portion **214** with an attachment lug **218** connects to the shaft **204** and is secured thereto by a shaft end piece **220**. A recoil spring **224** is positioned under compression on the telescoping shaft between the housing or receiver engagement portion and the spring stop. The assembly is inserted into the recoil spring assembly opening which is sized to allow freedom of movement of the spring and telescoping shaft, particularly to compress and expand. A forward end **228** of the shaft **204** is inserted in the aperture **155** on the handle intermediary member **154** and extends into the aperture on the spanning member **158** thereby effectively locking the handle assembly and bolt locking mechanism **93** in place in the bolt body.

Embodiments of bolt locking mechanism **93** in accord with the inventions herein are illustrated in FIGS. **5-7**, **15-22**B. In embodiments, the movable member **94** extends from the bolt body and is movable inwardly and outwardly which in a normal firing position of a firearm, is vertically. The movable member is movable from an extended position as shown in FIGS. **5**, **7**, **18**, **21**, **22**B, **30**, **31**, **35** to a retracted position as illustrated by FIGS. **19**, **23**B, **24**B, **32**, **33** and back and forth. In an embodiment as illustrated in FIGS. **18** and **19**, the vertically movable member **94** has an outward (shown also as upward) bias as provided by, for example, a coil spring **230** and a cam follower surface on one end. In another embodiment, such as shown in FIG. **21**, the movable member has cam follower surfaces on opposite ends. When the rimfire cartridge **119** is fired by impact with the firing pin with the rim **236** of the cartridge, the bolt assembly **48** cannot move rearwardly until the movable member is retracted. Since the force provided to retract the movable locking member is acting essentially at 90 degrees from the needed direction of retraction, there is a substantial force multiplication requirement of what is needed at the bolt to accomplish the retraction at the movable member. In the embodiment of FIGS. **18** and **19**, the spring force of the coil spring **230** can be adjusted to provide appropriate retraction resistance of the movable member to delay the retraction and blowback. The recoil spring assembly **151** directly engages the bolt body **82** in this embodiment. In embodiments, the movable member can be positioned in different locations on the bolt to interact with the cam surface on the housing adjacent thereto. Although embodiments in this application illustrate cooperation with the engagement or cam surface **67** on the ceiling **61**, the upper part of the receiver, interaction could also take place on the sides of the receiver or housing. More than one such movable member and cooperating cam surfaces can be utilized.

Referring to FIGS. **20** and **21**, a further embodiment which has the upward bias on the movable member **95** but the bias is provided through the ramp portion **159** that has a forward bias provided by a recoil spring coaxial with the axis of the firearm barrel. Arrow **231** indicates the force applied to the bolt assembly **48** by a fired cartridge, said force is transmitted to the movable member **94** through the bolt body **82**. In order for the bolt assembly to move rearwardly, the movable member needs to retract from the recess **66**. The inclined surface **67** provides the downward reactionary force to move the movable member **94** downwardly. Additional metal to metal frictional forces, indicated by the arrows **233** provide resistance to the downward movement.

Additionally, the movable member must push the ramp portion **159** rearward with respect to and within the bolt body **82** to retract. This is accomplished by way of the downward force on the ramp surface **161**. This rearward movement is "squeezing" the ramp portion or wedge out-of-the-way of the movable member. It can also be described as a reverse cam mechanism with the component which is configured as a cam follower pushing on the component that is configured as having the cam surface to move that component—the ramp portion. The resistance of the ramp portion to moving rearward is highly dependent upon angle **234**, the lesser the angle the more downward force, as indicated by arrow **238**, is needed.

Significantly, the carrier with the ramp portion moving forward is essentially has a reverse cam mechanism **235**. That is, what would be traditionally a cam follower, the lower surface **237** of the movable locking member **94** is forcing the movement of what would normally be the cam surface, the ramp portion **159**. And forcing it in a direction substantially normal to the force provided by the movable member and the ramp portion is biased against the movement by the recoil spring assembly **151**. This provides a great multiplication of the blowback resistance of the bolt over what would be provided in a simple blowback arrangement where the resistance to blowback is provided by the inertia of the mass of the bolt assembly and the resistance provided by the recoil spring and frictional resistance. This mechanism also provides a dramatic increase over the configuration of FIGS. **18** and **19**. The arrangement shown schematically in FIGS. **22**A-**25**B has been shown by the applicant, in necked rimfire cartridges, to provide a highly reliable semiautomatic cycling action. Such reliability has not been commercially seen previously in a semiautomatic rifle for necked rimfire cartridges. The incline angles **234**, **239** of the sliding surfaces can be adjusted to increase or decrease the force multiplication for blowback of the bolt assembly.

Referring to FIGS. **22**A-**25**B, the sequence of stages in recycling the firearm with such a delayed blowback configuration as described with reference to FIGS. **20** and **21** above is illustrated. FIGS. **22**A and **22**B shows the bolt assembly in an in-battery condition, ready to fire. The movable locking member **94** is engaged in the recess in the ceiling of the receiver. In FIGS. **23**A and **23**B, the rearward force provided by the firing of the cartridge has forced the locking member downwardly by pushing the ramp portion rearwardly against the bias of the spring **224**. In FIGS. **24**A and **24**B, the bolt is in the full retraction position, the spring is compressed and will return the bolt assembly to the in-battery position as illustrated by FIGS. **25**A and **25**B and urge the movable locking member **94** into the recess by way of the ramp portion **159**.

The firing pin blocking mechanism **95** is illustrated best in FIGS. **5-7**, **13-17**, **31-43**. In embodiments, the firing pin mechanism may be locked out in two ways, first by an interference with forward motion and secondly by way of removing the exposed striking end of the firing pin such that the hammer cannot strike it. The outwardly projectable movable member **94** is a blocking member with respect to this mechanism and function. The blocking member may have an inverted T-shaped opening **240** that cooperates with structure **242** on a forward portion **244** of the firing pin **162**. The wedge **248** as illustrated in FIGS. **31-36** represents the ramp portion **159** or cam surface of the blocking member carrier **158** and FIGS. **37**A-**40**B further illustrate the positioning of the firing pin **162** during different stages of operation. FIGS. **17**, **31**, **32**, **37**A, **37**B, **40**A, **40**B correspond to the in-battery position of the bolt assembly in a ready-to-fire mode with the blocking member at an elevated position on the ramp portion and with the outward engagement tip **250** or cam follower surface **96** of the movable member **94** engaged in the recess **66**. The movable blocking member **94** is thus in a non-blocking position and the firing pin is extending through the widest or largest portion of the opening, the non-blocking opening portion **249**, of the inverted T-shaped aperture. This allows the firing pin structure, specifically the stop portions **176**, **180**, to pass through unobstructed. The movable blocking member may have a bearing surface **252** including a tapered lead-in surface **254** on which the firing pin may rest or engage during forward and rearward motion. Similarly, the blocking member carrier **158** including the ramp portion **159** may have cut away portions **258** and bearing surfaces **260**. In this in-battery position, as best seen in FIGS. **17**, **37**A and **37**B, the rearward striking end **168** of the firing pin is exposed out of the bolt body **82** and the forward tip **164** is displaced from the cartridge head space **112** in the bolt body.

FIG. **32** illustrates the position of the firing pin with respect to the blocking member upon being struck by the hammer and impacting the cartridge. This generally is the furthermost forward position of the firing pin. The movable blocking member **94** is still engaged in the recess **66** in the ceiling of the receiver **34**. In FIGS. **33**, **38**A, and **38**B the force from the ignited cartridge has acted upon the bolt assembly driving same rearwardly as it forces the cam follower portion of the blocking member inwardly (downwardly), as indicated by arrow **264** in FIG. **32**, by way of the transition cam surface **67**. This inward (or downward) forces transmits the force downward on the ramp portion and due to the inclined surface engagement, forces the ramp portion, as indicated by arrow **266**, and the blocking member carrier

158 rearwardly within the bolt body and further blows the entire bolt assembly rearward against the resistance provided by the recoil spring **224**. With the blocking member moved downwardly as illustrated in FIG. **22**C, the firing pin now passes through the narrowed blocking portion **272** of the firing pin opening **240**. In the embodiment illustrated, the firing pin did not have time to retract after impacting the cartridge and a blocking portion **276** of the movable blocking member engages the rearwardmost reduced diameter or thinned portion **174**, of the firing pin, see FIG. **33**, and interferes by way of the more rearward stop portion **180**. Further rearward retraction of the firing pin continues, see FIG. **34**, as urged by the recoil spring **224**. At this stage, the firing pin is fully retracted within the bolt body as illustrated by FIGS. **38**A and **38**B, shielded from the hammer, and the bolt assembly proceeds to its full recoil position as shown in FIGS. **39**A and **39**B with the firing pin still completely enclosed in the bolt body. When the bolt assembly returns towards the in-battery position, movable blocking member **94** will transition, as illustrated by FIG. **35**, into the recess. In the fully seated position of the blocking member in the recess of FIGS. **36**, **40**A, and **40**B as illustrated, the firing pin is now in the non-block region of the opening, is exposed out the rearward end of the bolt body **82** and the firearm is ready to fire. FIG. **34** presents a first position for the movable member where the firing pin is blocked and FIG. **31** presents a second position where the firing pin is not blocked.

Referring to FIGS. **15**, **41**A, **41**B, **42**, and **43**, use of the manual handle **92** when the bolt assembly is in the in-battery position is illustrated. In that there is forward and backward clearance between the movable blocking member carrier **159** and the slot **148** in the bolt body **82**, the engagement of the cooperation between the blocking member **94** and the ramp portion may be manually effected. With the bolt assembly in the in-battery position, as illustrated in FIGS. **17** and **31**, **37**A, and **37**B, the manual handle may be grasped and urged rearwardly against the force of the recoil spring which is directly connected to the handle and carrier assembly. Referring to FIG. **41**, the manual handle may be moved from the original position **227**, shown by the dashed line, to the position of the solid lines. In an embodiment, this can be accomplished without taking the bolt assembly out of the in-battery position. The clearance **284**, see FIG. **41**A, is sufficient such that the ramp portion may be moved from the position where the movable blocking member is extended and on the upper portion of the inclined surface to the position where the blocking member is on the lower portion of the inclined surface without moving the entire bolt assembly. The tip of the blocking member then is no longer engaged with the recess in the ceiling of the receiver. Additionally, with the lowering of the blocking member, the forward thinned or reduced diameter portion of the firing pin is captured in the narrow portion of the opening in the blocking member as illustrated in FIG. **42**. Rearward movement of the handle may withdraw the cartridge from the chamber and with the firing pin locked as shown in FIG. **42**, striking of the exposed rearward end **168** of the firing pin by the hammer will restrict the forward motion of the firing pin to that shown in FIG. **42** which is insufficient for the firing pin to reach the headspace where the cartridge is seated in the bolt body. Continued rearward movement of the handle with take the bolt position to that as illustrated in FIGS. **24**A and **24**B, where, if a cartridge is in the bolt face, the cartridge can be ejected by the ejector **124**, shown in FIG. **4**. A cartridge in the magazine will be loaded as the bolt returns to the in-battery position. This sequence is utilized for loading the first cartridge from the magazine.

Referring to FIG. **44**, an embodiment is illustrated in which a movable member **294** has a pivot arm **296** and is pivotally connected to the bolt body **297** at a pivot point **298**. The movable member has opposing ends **304**, **304**, with sliding engagement surfaces **310**, **312**. The movable member may have a recess or opening, not shown, for the firing pin **316** shown by dashed lines. Rather than floating within the bolt body, this embodiment has the member attached thereto. The motion is in an arc rather than the linear movement of the floating embodiment. In other embodiments, the configuration of FIGS. **18** and **19** could utilize a pivotally connected movable member as well and the spring bias, could be between the pivot arm and bolt body, or could be attached to other structure.

Firearms with delayed blowback mechanisms are known and firearms with firing pin blocks are known. See for example U.S. Pat. Nos. 4,344,246; 1,737,974; 1,410,270; 6,782791; 3,857,325; 2,975,680; and 5,666,754. These patents are incorporated by reference for all purposes. Aspects of the instant application will be suitable for incorporation in known mechanisms.

Referring to FIGS. **45-60**, a firearm **1030** generally comprises a trigger assembly **1032**, a barrel **1034** mounted in a stock **1036** and connecting to a receiver **1037**. A firearm housing **1038** formed of the receiver **1037** and stock in this embodiment, engages and extends rearwardly from the barrel **1034** and houses a breech **1042** and the trigger assembly **1032**. The breech **1042** is above and forward of the trigger assembly **1032** and rearwardly of the barrel. The barrel **1034** has a body portion with a smaller outer diameter male threaded portion **1040** defining a firing chamber **1041** concentric about a barrel axis **1043**, the male threaded portion **1040** threadably engaging with a female threaded portion **1042** of the receiver **1037**. In one embodiment, the chamber is configured for necked cartridges, such as the .17 HSR and .17 WSM. A locking nut **1044** can threadably engage a larger outer diameter threaded portion **1046** of the barrel and tighten against the forward end **1048** of the receiver **1037**.

A bolt assembly **1052** is slidingly engaged within the receiver **1037** and includes a cartridge retraction mechanism **1051**, and a manual handle **1056**. A cycling spring assembly **1055** connects between the bolt assembly and the rearward end **1057** of the trigger assembly. A trigger guard **1056** extends from the housing **1038**.

The trigger assembly **1032** is depicted in detail and various views throughout the figures. The trigger assembly **1032** is housed within the firearm housing **1038** comprising primarily the stock **1036**. The trigger assembly **1032** has a trigger mechanism housing **1058** which receives a trigger component cluster **1059** as best shown in FIG. **49**A. The trigger component cluster **1059** are generally movable components and pivot about shafts that are supported by the firearm housing **1038**. The cluster **1059** is depicted in various views without the housing **1038** for purposes of clarity. The firearm housing **1038** is advantageously formed from injection molding polymers and may have specific metal inserts therein for reinforcement, for example at the rearward projection **1060** that is inserted in a cooperating aperture **1061** in the rearward end of the receiver **1037**.

Referring to FIGS. **49**A-**56**, within the trigger mechanism housing **1058**, the trigger component cluster **1059** generally includes a hammer **1082**, a firing trigger component **1084**, a safety trigger component **1086**, an arrestor **1088**, and a manual safety mechanism **1090**. The hammer **1082** includes a head portion **1092** and a cam portion **1094** having separated by a stem portion **1096**. The cam portion **1094** defines an aperture **1098** that is mounted to and rotates about a bushing **1100** and shaft **1101** to define a hammer pivot **1102** that actuates about a rotational axis **1104**. In one embodiment, the cam portion **1094** further includes an arcuate cam surface **1105** and a sear engagement portion **1106**, the sear engagement portion **1106** having a radially extending bearing face **1108**. The cam portion **1094** can also define a flat **1110** that extends at an angle θ from the bearing face **1108**. In one embodiment, the angle θ is an obtuse angle. The hammer **1082** is also coupled with a biasing element **1112** which, in some embodiments, is a rotational spring **1114** (FIGS. **55** and **58-66**) that is rotated about and coupled to the hammer pivot **1102** with the free ends engaged, for example, with the trigger mechanism housing **1058**. The hammer **1082** can also include a capture feature **1116**. In various embodiments, the capture feature **1116** includes an engagement surface **1115**. A squared loop **1117** in the rotational spring **1114** can provide space at the projection for engagement of the projection with the safety trigger component, discussed below.

As best seen in FIGS. **50**, **51**, **52**, **53** and **56**, the firing trigger component **1084** includes a finger hook portion **1122** and a sear portion **1124**, the sear portion **1124** having a sear surface or cam engagement surface **1140** cooperating with and being configured to engage the sear engagement portion **1106** and cooperating surface **1108** of the hammer **1082**. The firing trigger component **1084** can be mounted to a trigger pivot **1126** configured as a shaft or pin and defining a rotational axis **1128** and extending from the trigger mechanism housing **1058** along the rotational axis **1128**. In some embodiments, the firing trigger component **1084** further defines a slot **1132** that extends into the finger hook portion **1122** and lies on a plane that is substantially perpendicular to the rotational axis **1128**. The firing trigger component **1084** can also include an extended portion **1134** that is engaged with a firing trigger return spring **1136** that biases finger hook portion **1122** of the firing trigger component **1084** in the forward direction **1081**. The return spring **1136** may be engaged with a ledge or flange portion **1137** of the trigger mechanism housing (FIGS. **48**, **49**A **49**B, **50** and **52**).

In some embodiments, the firing trigger component **1084** includes a cam engagement surface **1140** that engages the arcuate cam surface **1105** of the hammer **1082**.

The safety trigger component **1086** can include a finger hook portion **1142** and can be pivotally mounted to the trigger pivot **1126**. In various embodiments, the finger hook portion **1142** of the safety trigger component **1086** is a flat structure, formed from, for example, sheet or plate, that is disposed in the slot **1132** of the finger hook portion **1122** of the firing trigger component **1084**. The finger hook portion **1122** of the safety trigger component **1086** can also include an aperture **1144**. The aperture **1144** can be utilized for insertion of a pin or lock, effectively preventing movement of the trigger hook portion particularly with respect to the hook portion of the firing trigger component. As discussed further below, this prevents the firing trigger component **1084** from being actuated.

In one embodiment, the safety trigger component **1086** includes a catch portion **1146** that is laterally adjacent to the hammer **1082**. The catch portion **1146** can resemble an inverted "J" shape, for example as depicted in FIGS. **46** and **47**. The safety trigger component **1086** can also include an extended portion **1148** that is engaged with a safety trigger component return spring **1152**. The return spring **1152** is attached to the ledge portion **1137** of the trigger mechanism housing configured as a ledge. In one embodiment, the extended portion **1148** of the safety trigger component **1086** includes an arm **1154** that extends out of the slot **1132** and wraps over and partially around the extended portion **1134** of the firing trigger component **1084**, as best seen in FIGS. **49**A **51**, **52** and **53**. A spring receiving member **1155** shaped as a projection receives the safety trigger return spring **1152**.

Functionally, the safety trigger component return spring **1152** exerts a return force on the extended portion **1148** of the safety trigger component **1086** urging the finger hook portion **1142** of safety trigger component **1086** to be rotated to a full forward position within the slot **1132** of the firing trigger component **1084**. In this unactuated or default orientation, the catch portion **1146** is positioned so that the catch portion **1146** is in a rotational path **1162** (FIG. **58**) through which the capture feature **1116** of the hammer **1082** travels during firing and obstructs the hammer **1082**. Accordingly, the catch portion **1146** intercepts the capture feature **1116** of the hammer **1082** if the catch portion **1146** of safety trigger component **1086** has not first been rotated out of the rotational path **1162**. Hence, the safety trigger component **1086** provides an additional safety mechanism that helps prevent discharge of the firearm **1030** in the event of an unintentional release of the hammer **1082**—for example, during an impact event where the weapon becomes jarred to the extent that the sear portion **1124** of the firing trigger component **1084** slips off the sear engagement portion **1106** of the hammer **1082**.

During such an impact event, the safety trigger component **1086** may undergo rotational displacement that is commensurate with the rotational displacement of the firing trigger component **1084**. However, in various embodiments, the rotational displacement required to rotate the catch portion **1146** out of the rotational path **1162** of the capture feature **1116** of the hammer **1082** is substantially greater than the rotational displacement required for the sear portion **1124** of firing trigger component **1084** to disengage the sear engagement portion **1106** of the hammer **1082** (see discussion below). Accordingly, the safety trigger component **1086** will generally still perform the function of intercepting the hammer **1082** even if the safety trigger component **1086** undergoes the same or even somewhat more rotational displacement than the firing trigger component **1084** in an impact event.

In the depicted embodiments, the capture feature **1116** is a lateral projection that extends laterally outward from the hammer **1082** in a direction parallel to the rotational axis **1104**, for capture by the inverted "J" or other concavity defined by the catch portion **1146**. In other embodiments, the capture feature **1116** can comprise a notch formed in the hammer **1082**, and the catch portion **1146** can include a projection that is captured within the notch (not depicted).

Referring to FIGS. **57** through **59**, an operation sequence of the hammer **1082**, the firing trigger component **1084**, the safety trigger component **1086**, and the bolt assembly **1052** from a fully cocked configuration **1180** to a triggered configuration **1182** is depicted in one embodiment of the disclosure. The FIGS. **57-60** depict the hammer **1082**, firing trigger component **1084**, and safety trigger component **1086** at a mid-plane of the slot **1132**, with various appurtenances removed for clarity of illustration.

In the fully cocked or "battery" configuration **1180** (FIG. **57**), the sear portion **1124** of the firing trigger component **1084** is in forced engagement with the sear engagement portion **1106** of the hammer **1082**, the forced engagement being exerted by the biasing element **1112**. The respective finger hook portions **1122** and **1142** of the firing trigger component **1084** and the safety trigger component **1086** are held in a forward most orientation by the respective return springs **1136** and **1152** (FIGS. **50**, **52**, **53**). In the fully cocked configuration **1180**, the bolt assembly **1052** is also in a firing position within the breech **1042**, with a firing pin **1054** exposed and outwardly extending relative to a rearward end **1183** of the bolt assembly **1052**. In one embodiment, the firing pin **1054** is substantially parallel to but offset from the barrel axis **1043** to facilitate firing of rimfire cartridges. Also in the fully cocked configuration **1180**, a front edge **1184** of the safety trigger component finger hook portion **1142** extends distal to a front edge **1186** of the firing trigger component finger hook portion **1122**.

An actuation force **1192** is applied to the front edge **1184** of the safety trigger component finger hook portion **1142** (FIG. **58**), for example by a squeezing motion applied by a finger of a user. The actuation force **1192** causes the safety trigger component **1086** to rotate about the trigger pivot **1126**, so that the catch portion **1146** is rotated out of the rotational path **1162** of the capture feature **1116**, thereby clearing the hammer **1082** for an unobstructed rotation to the firing pin **1054**. In the FIG. **58** depiction, the safety trigger component **1086** is progressing toward a firing position, while the firing trigger is in a battery position.

The actuation force **1192** then engages the firing trigger component **1084**, thereby causing the firing trigger component **1084** and the safety trigger component **1086** to rotate effectively simultaneously about the trigger pivot **1126** and into firing positions. The rotation of the firing trigger component **1084** causes the sear portion **1124** to rotate away from the hammer **1082** and slide radially outward from the hammer pivot **1102** along the sear engagement portion **1106**. When the sear portion **1124** slides off the sear engagement portion **1106**, the hammer **1082** is released and swings into contact with the firing pin **1054**, thereby establishing the triggered configuration **1182** where both the safety trigger component **86** and the firing trigger component **1084** are in a firing position (FIG. **59**).

The positions of respective finger hook portions **1122** and **1142** of the firing trigger component **1084** and the safety trigger component **1086** for both the fully cocked configuration **1180** and the triggered configuration **1182** are presented in FIG. **59**, with the positions from the fully cocked configuration **1180** being presented in phantom. Angular displacements α and β of the safety trigger component **1086** and the firing trigger component **1086**, respectively, are also overlaid onto FIG. **59**. By this illustration and for this embodiment, the angular displacement α of the safety trigger component **1086** in transitioning from the fully cocked configuration to the triggered configuration is about three times greater than the angular displacement β of the firing trigger component **1084**. As such, the safety trigger component **1086** will generally still perform the function of intercepting the hammer even if the safety trigger component **1086** undergoes the same or even somewhat more rotational displacement than the firing trigger component **1084** in an impact event.

Referring to FIG. **60**, the functionality of the safety trigger component **1086** during an abnormality such as an impact event is further illustrated in an embodiment of the disclosure. Consider an impact event where inertial forces cause a dynamic load **1188** on the respective finger hook portions **1122** and **1142** of the firing trigger component **1084** and the safety trigger component **1086**, such that both finger hook portions **1122** and **1142** are rotationally displaced by the angular displacement β required to release the hammer **1082**. At the angular displacement β, the catch portion **1146** is still operational within the rotational path **1162** of the capture feature **1116**, and still functions to arrest the hammer **1082** and prevent discharge of the firearm **1030**.

Referring again to FIGS. **48** through **54**, and **56**, the trigger assembly **1032** includes the manual safety mechanism **1090** conventionally positioned forward of the firing trigger. The safety mechanism **1090** includes a safety bar **1194** with exposed push buttons **1195**, **1196** on each end, a shaft **1197** integral with one of the push buttons **1195**, **1196** for aligning and securing the safety mechanism components together, and a rotatable blocking member **1200**. A pin **1198** may extend through apertures **1199**, **1201** in the shaft **1197** and end button **1196** to secure the manual safety mechanism **1090**. The blocking member **1200** can include a lever portion **1202** that projects radially outward from an arcuate base portion **1204**. The arcuate base portion **1204** rotates freely about a blocking member pivot **1206** defined by the shaft **1197**. In one embodiment, a notch or recess **1208** is formed on the arcuate base portion **1204** to provide a non-blocking position for an engagement tab **1209** proximate the sear portion **1124** of the trigger component. The manual safety mechanism **1090** is laterally slidable within the trigger mechanism housing **1058** in apertures **1210**, **1213** on opposing sides of the housing **1058**.

The safety trigger component **1086** can include a fork **1211** comprising a pair of protrusions **1212***a* and **1212***b* that contact the blocking member **1200**. The firing trigger component **1084** can include an underside **1214** against which the lever **1202** of the blocking member **1200** registers. In the depicted embodiment, the underside **1214** defines a recess **1215** within which the lever **1202** registers The firing trigger component **1084** can further include a projection **1216** that is proximate the arcuate base portion **1204** of the blocking member **1200**.

Referring to FIGS. **61** through **63**, operation of the blocking member **1200** during discharge of the firearm **1030** is depicted in an embodiment of the disclosure. In the fully cocked configuration **1180** (FIG. **53**), the lever portion **1202** of the blocking member **1200** extends between the protrusions **1212***a* and **1212***b* and is engaged or nearly engaged within the underside **1214** of the firing trigger component **1084**. The protrusion **1212***b* of the safety trigger component **1086** maintains the blocking member **1200** in engagement/near engagement with the firing trigger component **1084**, thereby preventing the firing trigger component **1084** from rotating away from the hammer **1082**. Also in the fully cocked configuration **1180**, the arcuate base portion **1204** of the blocking member **1200** can also interfere with the projection **1216** of the firing trigger component **1084**, further preventing actuation of the firing trigger component **1084**.

During actuation of the safety trigger component **1086**, the protrusion **1212***a* rotates against blocking member **1200**, causing the lever portion **1202** to rotate away from the underside **1214** of the firing trigger component **1084**. The rotation of the blocking member **1200** also causes the recess **1208** of the arcuate base portion **1204** to rotate into alignment with the projection **1216** of the firing trigger component **1084** (FIG. **54**). During continued actuation of the safety trigger component **1086** and subsequent actuation of the firing trigger component **1084**, the lever portion **1202** has now been removed as an obstacle to rotation of the firing trigger component **1084** (FIG. **55**), and the recess **1208** now accommodates the projection **1216** of the firing trigger component.

Accordingly, when the firearm **1030** is in the fully cocked configuration, the safety trigger component **1086** controls the orientation of the blocking member **1200**. As the safety trigger component **1086** is actuated, the blocking member

1200 is oriented so as not to pose an obstruction to the firing trigger component 1084, freeing the firing trigger component 1084 for rotation away from the hammer 1082 and subsequent discharge of the firearm 1030.

Functionally, in the fully cocked configuration 1180, if an actuation force or "pull" is exerted on the firing trigger component 1084 but somehow not exerted on the safety trigger component 1086, the blocking member 1200 will maintain engagement with the firing trigger component 1084, thereby preventing rotation of the firing trigger component 1084 and subsequent discharge of the firearm 1030. Thus, in one embodiment, the blocking member 1200 can provide a redundant or additional safety mechanism against accidental discharge of the firearm 1030. Instead of relying solely on the friction between the sear portion 1124 and the sear engagement portion 1106, the blocking member 1200 provides a positive blocking force that helps prevent disengagement of the sear and the sear engagement portions 1124 and 1106 in an impact event. Moreover, the lever portion 1202 engaging the recess in the trigger component prevents the pivoting of the component about the pivot. In some embodiments, the blocking member 1200 can be the sole safety mechanism; that is, the blocking member 1200 is utilized without the catch portion 1146 instead of in addition to the catch portion 1146.

Referring to FIGS. 64 through 66, restoring the trigger assembly 1032 from the triggered configuration 1182 to the fully cocked configuration 1180 (referred to herein as "cocking") is depicted in an embodiment of the disclosure. After discharge of the firearm 1030, the projection 1216 of the firing trigger component 1084 is seated in the recess 1208, held in place by the cam portion 1094 of the hammer 1082 (FIG. 64). The seating of the projection 1216 in the recess 1208 prevents rotation of the blocking member 1200; that is, in the triggered configuration 1182, the orientation of the blocking member 1200 is not controlled by the safety trigger component 1086 (as is the case in the fully cocked configuration 1180), but instead is controlled by the firing trigger component 1084 and hammer 1082. Accordingly, the blocking member 1200 now acts against protrusion 1212*b* to hold the safety trigger component 1086 in a pitched orientation, wherein the catch portion 1146 is rotated away from the rotational path 1162 of the capture feature 1116.

The bolt assembly 1052 is motivated in the forward direction 1080 by a force 1222, imparted, for example, manually by a gunman or by a blow back mechanism. This motivation causes the bolt assembly 1052 to rotate the head portion 1092 of the hammer 1082 in the forward direction 80, which further causes the cam portion 94 to rotate on the cam engagement surface 1140. The cam engagement surface 1140 is maintained in contact with the cam portion 1094 by a return force 1224 imparted on the firing trigger component 1084 by the firing trigger return spring 1136.

As the head portion 1092 of the hammer 1082 is rotated in the forward direction 1080, the capture feature 1116 is rotated below the hook of the catch portion 1146 (FIG. 57), while the cam portion 1094 of the hammer 1082 maintains the interlock between the firing trigger component 1084 and safety bar 1200 (and therefore the pitched orientation of the safety trigger component 1086).

At some point after the capture feature 1116 of the hammer 1082 is rotated below the hook of the catch portion 1146, the arcuate cam surface 1105 of the cam portion 1094 rotates off the cam engagement surface 1140 (FIG. 58). At this point, the arcuate cam surface 1105 of the cam portion 1094 releases the firing trigger component 1084. The firing trigger component 1084, motivated by the return force 1224 generated by the firing trigger return spring 1136, then rotates (counterclockwise in FIG. 58) so that the cam engagement surface 1140 is brought into contact with the flat 1110 of the cam portion 1094; the sear portion 1124 of the firing trigger component 1084 is brought adjacent to the sear engagement portion 1106 of the hammer 1082. The release of the firing trigger component 1084 by the arcuate cam surface 1105 also causes the projection 1216 of the firing trigger component 1084 to become unseated from recess 1208 of the blocking member 1200. Control of the orientation of the blocking member 1200 is thereby transferred to the safety trigger component 1086, which, propelled by the return force 1224, rotates the blocking member 1200 (clockwise in FIG. 66) into the underside 1214 of the firing trigger component 1084.

Upon withdrawal of the bolt assembly from contact with the hammer 1082 and into the firing position, the fully cocked configuration 1180 of the firearm 1030 is restored (e.g., FIG. 61), with the blocking member 1200 preventing actuation of the firing trigger component 1084 that is independent of actuation of the safety trigger component 1086, and the catch portion 1146 poised to intercept the hammer 1082 in case of unintentional release of the hammer 1082.

In one embodiment, and again in reference to FIGS. 48 through 54 and 56, the blocking member 1200 is part of a manual safety mechanism 1230 that can be translated with the blocking member 1200 laterally within the trigger mechanism housing 1058 along a blocking member axis 1234. When part of the manual safety mechanism 1230, the lever 1202 of the blocking member 1200 can be selectively engaged with a stop 1236 (best seen in FIGS. 49B and 50) that extends from the interior surface 1044 of the trigger mechanism housing 1058 along the right side wall 1237 of the trigger mechanism housing 1058. In the embodiment illustrated, when the manual safety mechanism 1230 is pushed in one direction (e.g., to the right in the depicted embodiments), the firearm 1030 is configured in a "safety mode," wherein the blocking member lever 1202 is prevented from rotating out of the blocking position by the ramp or stop 1236.

When the manual safety mechanism 1230 is pushed in an opposite direction (e.g., to the left in the depicted embodiments), the firearm is configured in a "firing mode," wherein release of the sear portion 1084 of the firing trigger component 1084 from the sear engagement portion 1106 of the hammer 1082 is enabled. In the firing mode, the lever portion 1202 is displaced off of the stop 1236, enabling rotation by the fork 1211 of the safety trigger component 1086 and rotation the lever portion 1202 out of the blocking position with the underside 1214 of the firing trigger component 1084. The lever 1202 can be sized widthwise such that, during lateral movement of the blocking member 1200, the lever maintains engagement of the safety trigger fork 1211. Also, the lever 1202, when engaged with the underside 1214 on the lower side of the firing trigger component 84, can maintain blockage and/or engagement with the underside 1214 during lateral actuation. Engagement with the underside 1214 is lost only upon the rotation of the blocking member 1200.

It is further noted that aspects of the embodiments depicted in FIGS. 61 through 66 may be suited for automatic operation. (Herein, "automatic operation" is characterized as the continuous, round after round discharge of ammunition as long as the firing trigger component 1084 is depressed.) For the embodiments of FIGS. 61 through 66, as long as the triggers 1084 and 1086 are held in the firing position (depicted in FIG. 63), the sear portion 1124 of the firing trigger component **1084** will not be brought into engagement with the sear engagement portion **1106** of the hammer **1082**, and the catch portion **1146** will not obstruct the hammer **1082** in either rotational direction. Accordingly, certain aspects of the embodiment of FIGS. **51** through **58** can be utilized in an automatic firearm.

Referring to FIGS. **67** through **69**, an arresting mechanism **1260** that facilitates semi-automatic operation (as opposed to automatic operation) is depicted in an embodiment of the disclosure. (Herein, "semi-automatic operation" is characterized by the automatic reloading of the firearm **1030**, but the requirement to release and re-actuate the triggers **1084** and **1086** to initiate firing.)

In one embodiment, the arresting mechanism **1260** involves interaction of at least four components: the bolt assembly **1052**, the hammer **1082**, the firing trigger component **1084**, and an arrestor **1088**. The arrestor **1088** is pivotally mounted within the housing **1038** and distal to the hammer **1082**. In one embodiment, the arrestor **88** includes a claw portion **1264** and a rocker arm portion **1266**. The claw portion **1264** can include a rounded head portion **1268** and a radiused nose **1272**. An arrestor return spring **1274** can be operatively coupled to the arrestor **1088**. In one embodiment, the arrestor **1088** is pivotally mounted to the trigger pivot **1126**.

In various embodiments, the arresting mechanism **1260** can include a cavity **1282** formed in the head portion **1092** of the hammer **1082**, the cavity **1282** and head portion **1092** further defining a lip portion **1284**. In one embodiment, the firing trigger component **1084** includes a lateral protrusion **1286** that is part of the arresting mechanism, the lateral protrusion **1286** being positioned to engage the rocker arm portion **1266** of the arrestor **1088**.

In one embodiment, the arrestor **1088** is configured and positioned so that the claw portion **1264** is engageable with the lip portion **1284** of the cavity **1282** when the hammer **1082** is hyperextended in the forward direction **1080**. Herein, the hammer **1082** is considered "hyperextended" when the head portion **1092** of the hammer **1082** is displaced to be forward to where the head portion **1092** is located when in the fully cocked configuration **1180**.

Referring to FIGS. **70** through **75**, operation and function of the arresting mechanism **1280** in a scenario where the triggers **1084** and **1086** become or remain actuated during the cocking of the firearm **1030** is depicted in an embodiment of the disclosure. Functionally, the arresting mechanism **1260** captures the hammer **1082** and prevents the hammer **1082** from automatically re-firing. To more closely resemble the views presented in FIGS. **67** through **69**, the FIGS. **70** through **75** are presented in an opposing side view relative to the views of FIGS. **61** through **66**. Also, for illustrative clarity, the biasing element **1112**, as well as the various return springs **1136**, **1152** and **1274**, are not presented in FIGS. **70** through **75**, though they may be present in certain embodiments. Also for illustrative clarity, only the components of the arresting mechanism **1260** (i.e., the bolt assembly **1052**, the hammer **1082**, the firing trigger component **1084**, and an arrestor **1088**) are depicted in FIGS. **70** through **72**.

When an actuation force **1292** is applied to the triggers **1084** and **1086**, the lateral protrusion **1286** of the firing trigger component **1084** is pitched in the distal direction **1081**. The arrestor **1088**, being biased by the arrestor return spring **1274**, follows the firing trigger component **1084**, being stopped by the lateral protrusion **1286**. When the firing trigger component **1084** is depressed, the lip portion **1284** of the cavity **1282** encounters the rounded head portion **1268** and/or radiused nose **1272** of the claw portion **1264** as the head portion **1092** of the hammer **1082** is rotated in the forward direction **1080** during cocking of the firearm **1030** (FIG. **70**). The interaction between the lip portion **1284** and the rounded head portion **1268**, radiused nose **1272** of the claw portion **1264** to rotate slightly in the forward direction **1080**, such that the rocker arm portion **1266** rotates off the lateral protrusion **1286** of the firing trigger component **1084** (FIG. **71**). As the head portion **1092** of the hammer **1082** becomes hyperextended, the lip portion **1284** slips past the radiused nose **1272** of the claw portion **1264**, the arrestor **1088** is rotated so that the rocker arm **1266** is again in engagement with the lateral protrusion **1286** of the firing trigger component **1084**, motivated by a return force **1294** (FIG. **72**) generated by the arrestor return spring **1274**. The rotation causes the claw portion **1264** to rotate at least partially into the cavity **1282**.

The bolt assembly **1052** then retracts back into the firing position, becoming disengaged from the hammer **1082** (FIG. **73**). The disengagement causes the head portion **1092** of the hammer **1082** to rotate in the distal direction **1081** until the lip portion **1284** of the cavity **1282** is hooked by an underside **1296** of the claw portion **1264**. The arresting mechanism **11260** remains in equipoise as long as the firing trigger component **1084** remains in the actuated position. In this way, the arresting mechanism **1260** captures the hammer **1082** and prevents the hammer **1082** from automatically re-firing.

In one embodiment, upon removal of the actuation force **1292** (e.g., when the gunman removes his finger from the firing trigger component **1084**), the return force **1228** of the firing trigger return spring **1136** causes rotation of the firing trigger component **1084** so that the lateral protrusion **1286** of the firing trigger component **1084** is rotated upwards (clockwise in FIG. **74**). The lateral protrusion **1286** causes the rocker arm **1266** of the arrestor **1088** to also rotate upward, thereby decoupling the lip portion **1284** of the cavity **1282** from the underside **1296** of the claw portion **1264**. The lip portion **1284** of the hammer **1082** then slips past the radiused nose **1272** of the claw portion **1264**, being motivated by the biasing element **1112**, thereby releasing the hammer **1082** from the arrestor **1088**.

The rotation of the firing trigger component **1084** upon removal of the actuation force **1292** also causes the cam engagement surface **1140** to come into contact with the flat **1110** of the cam portion **1094**, which brings the sear portion **1124** of the firing trigger component **1084** proximate and adjacent to, but not in contact with, the sear engagement portion **1106** of the hammer **1082** (FIG. **74**). Upon release of the hammer **1082** from the arrestor **1088**, the head portion **1092** of the hammer **1082** further rotates in the distal direction **1081**, until the bearing face **1108** of the sear engagement portion **1106** is fully registered against the sear portion **1124** of the firing trigger component **1124** (FIG. **75**). The trigger assembly **1032** is then in the fully cocked configuration **1180**.

It is further noted that, in various embodiments, if the firing trigger component **1084** is not actuated when the hammer **1082** reaches the hyperextended position, the arrestor **1088** is not in a position to engage and/or secure the lip portion **1284** of the hammer **1082**. Accordingly, the arrestor **1088** does not substantially interfere with the cocking operation if the firing trigger component **1084** is not actuated.

The barrel and receiver may be conventionally manufactured from steel. In various embodiments, other metals may be used. The components of the trigger assembly cluster are generally conventionally formed from steel or other metals.

In some instances, polymers may replace some components. For example the trigger mechanism housing may be made from polymers and composite materials. Metal inserts may be used for particular areas requiring high strength such as attachment locations. See projection **1060** and the trigger guard **1056** (see FIGS. **49**A and **49**B). Also, see FIG. **47** the polymer access cover **1290** has a metal insert **1291** for strength and providing the catch surfaces. The polymer may be overmolded over the insert capturing the insert. The stock can be formed from polymers or wood or composite materials.

Referring to FIGS. **76** and **77**, a trigger pull adjustment mechanism **1300** is depicted in an embodiment of the disclosure. The trigger pull adjustment mechanism **1300** comprises an adjustable firing trigger return spring **1302** disposed in place of the firing trigger return spring **1136** (as depicted, for example, in FIG. **54**) and operatively coupled to the ledge portion **1137** and the firing trigger component **1084** to exert a separating force therebetween. This separating force constitutes a component of the pull or actuation force required to actuate the firing trigger component **1084** for releasing the hammer **1082**.

In the depicted embodiment, the adjustable firing trigger return spring **1302** includes an upper portion **1304** and a lower portion **1306** spiral wound about a spring axis **1308**. A transition segment **1312** can be formed in the lower-most spiral **1314** of the upper portion **1304**, the transition segment **1312** passing through the adjustable firing trigger return spring **1302** proximate the spring axis **1308**. In one embodiment, the transition segment **1312** is substantially linear over a portion thereof. In the way, the transition segment **1312** obstructs what would otherwise be a clear passage through the adjustable firing trigger return spring **1302**. The upper and lower portions **1304** and **1306** can be of different diameter, as depicted. Also in the depicted embodiment, the upper portion **1304** terminates with a tail portion **1316** that is substantially concentric with the spring axis **1308**. The ledge portion **1137** can define a mounting hole **1318** within which the tail portion **1316** is mounted in assembly.

In assembly, the lower portion **1306** of the adjustable firing trigger return spring **1302** is firmly seated within a through-hole **1322** defined on the firing trigger component **1084**. The firm seating of the lower portion **1306** within the through-hole **1322** can be accomplished by an interference fit between an inner wall **1324** of the through-hole **1322** and the lower portion **1306** of the spring **1302** as wound. The interference fit provides a high degree of friction between the inner wall **1324** of the through-hole **1322** and the lower portion **1306** of the spring **1302**, thereby fixing the compressed length of the spring **1302**. In this embodiment, while the friction is sufficient to maintain the compressed length **1302** of the spring when the firearm **1030** is in the fully cocked configuration **1180** (i.e., prior to actuation of the firing trigger component **1084**), the spring **1302** In one embodiment, the through-hole **1322** is tapered to augment the seating operation during assembly and rotation of the spring **1302** during an adjustment.

Referring to FIG. **78**, an adjustment tool **1330** for rotating the adjustable firing trigger return spring **1302** is depicted in an embodiment of the disclosure. The adjustment tool **1330** includes a shaft portion **1332** with a slot **1334** defined on one end thereof. A diameter **1336** of the shaft portion **1332** is dimensioned to readily pass through the interior of the lower portion **1306** of the spring **1302**. A width **1338** of the slot **1334** is dimensioned to receive the transition segment **1312** of the spring **1302**. Optionally, the adjustment tool **1330** includes a handle portion **1339** disposed proximate the end of the adjustment tool **1330** that is opposite the slot **1334**.

Referring to FIG. **79**, adjustment of the trigger pull adjustment mechanism **1300** is depicted in an embodiment of the disclosure. In the depicted embodiment, access passages **1342** are formed in the trigger guard **1056**, sized to allow passage of the shaft **1332** of the adjustment tool **1330**. The adjustment tool **1330** is inserted through the access passages **1342** and the lower portion **1306** of the adjustable firing trigger return spring **1302** and brought into contact with the transition segment **1312**. The adjustment tool is rotated and pushed against the transition segment so that the slot **1334** is aligned with and accepts the transition segment **1312**. With the transition segment **1312** seated within the slot **1334**, the adjustment tool **1330** is rotated to overcome the friction between the lower portion **1306** and the inner wall **1324** of the through-hole **1322**, thereby changing the compressive force of the spring **1302** when in the battery position. By increasing the compression of the spring **1302**, the restorative force generated by the spring **1302** is increased, thereby increasing the pull required to actuate the firing trigger component **1084**; by decreasing the compression of the spring **1302**, the restorative force generated by the spring **1302** is decreased, thereby decreasing the pull required to actuate the firing trigger component **1084**. The friction between the lower portion **1306** and the inner wall **1324** of the through-hole **1322** is sufficient to maintain the adjusted compression of the spring **1302** during operation of the firearm **1030**.

Accordingly, the disclosed trigger pull adjustment mechanism **1300** accomplishes adjustment of the trigger pull with fewer components and with reduced machining complexity. For example, conventional trigger pull adjustments utilize an additional set screw that requires a threaded hole for the compression adjustment. The trigger pull adjustment mechanism **1300** eliminates the need for these components and attendant complexity.

Other adjustable trigger mechanisms can be implemented instead. Such mechanisms are illustrated, for example, in U.S. Pat. No. 6,553,706, owned by the owner of this application, the disclosure of which is hereby incorporated reference herein in its entirety except for express definitions and patent claims contained therein. See also U.S. Pat. Nos. 8,220,193 and 8,250,799, the disclosures of which are hereby incorporated reference herein in their entirety except for express definitions and patent claims contained therein.

Referring to FIGS. **80** and **81**, a conventional rotating claw extractor **2020** operatively coupled to a bolt **2021** is depicted. The extractor **2020** rotates into contact with a shell casing **2022** having a case rim **2024** and a case wall **2026**, often making contact the case wall **2026** (FIG. **80**). In this position, the extractor **2020** exerts no force directly against the case rim **2024**. During extraction, a face **2028** of the bolt **2021** moves away from the shell casing **2022** until the extractor **2020** contacts the rim **2024**. Positive extraction is realized because the extractor **2020** exhibits a force on the case rim **2024**.

However, due to the size and shape of cartridges such as rim fire cartridges and in particular high powered rim fire cartridges, ejection can be problematic, for example in semi-automatic firearms. Ejection can become compromised because once the shell casing **2022** is extracted from the firing chamber it is not in static equilibrium and is no longer stable (FIG. **81**). That is, positive axial force is exerted asymmetrically, on only one portion of the case rim **2024**. The natures of the forces exerted on the shell casing **2022** are further complicated by dimensional uncertainties due to the manufacturing tolerances of the shell casing as well as the generally small dimensions. If these manufacturing tolerances cause the contact edge of the extractor **2020** to rotate into the headspace of the bolt face **2028**, the clearance can be inadequate for the extractor **2020** to secure the case rim **2024**. Additionally, if the contact edge of the extractor **2020** is near the headspace, feeding problems can occur as the case rim **2024** may get bound on the extractor **2020**. If dimensional uncertainties of the shell casing **2022** due to manufacturing tolerances cause the extractor to be displaced away from the headspace, the cartridge will again become unstable, as depicted in FIG. **81**.

It is further noted while other portions of the case rim **2024**, particularly portions that are diametrically opposed to the contact region of the rotating claw **2020**, can also be subject to an axial force, these axial forces rely on friction that results from radial counter forces exerted on the case rim **2024**. The frictional forces can be inconsistent, particularly when the surfaces involved are oiled, as is common practice with well-maintained firearms, or there is a buildup of discharge residue.

Referring to FIGS. **82** through **86**G a firearm **30** utilizing an extraction mechanism **2032** for extraction of spent cartridge casings therefrom is depicted in an embodiment of the disclosure. The firearm **2030** is a hand-held device that includes a barrel assembly **2034** mounted in a stock **2035** and operatively coupled to a receiver **2036**. The barrel assembly **2034** includes a barrel **2038** with a firing chamber **2042**, a breech **2044**, and a bolt assembly **2046** slidingly engaged within the breech **2044**. A trigger assembly **2048** is operatively coupled with the bolt assembly **2046**.

Various components of the bolt assembly **2046** are part of the extraction mechanism **2032**. The extraction mechanism **2032** includes a bolt **2052** having a bolt face **2054** at a distal end **2053** and a lower face **2055**. A recess **2058** is defined on the bolt face **2054**. In various embodiments, the structure defining the recess **2058** includes an undercut portion **2087** that extends distally to a ledge portion **2086**, the ledge portion **2086** having an arcuate segment **2060** that arcs tangentially about a central axis **2056** that is normal to the base surface **2072**. (Herein, an "axis" extends indefinitely in two opposing directions, and is not bound lengthwise by the object or feature that defines the axis.)

In one embodiment, the arcuate segment **2060** defines the location of the central axis **2056** on the base surface **2072**, the arcuate segment **2060** of the ledge portion **2086** being at a constant radius R from the central axis **2056**. The bolt **2052** being translatable parallel to the central axis **2056**. The recess **2058** can extend through a lateral periphery **2062** of the bolt **2052**, effectively defining a channel **2064** that extends along a channel axis **2066** and defining a channel opening **2068** at the lateral periphery **2062**. The recess **2058** can be bounded proximally by a base surface **2072** on the bolt face **2054**. The base surface **2072** is substantially normal to the central axis **2056**. The bolt assembly **2046** can further include a retractable anchoring bar **2070** that extends away from the central axis **2056** through an aperture **2071** formed in the bolt **2052**.

The bolt **2052** can also include structure defining a first lateral bore **2074** and a second lateral bore **2076** proximate the bolt face **2054**, the second lateral bore **2076** being proximal (rearward) to the first lateral bore **2074**. An extractor channel **2078** can be formed on the distal (forward) end portion **2053** of the bolt **2052**, the extractor channel **2078** extending parallel to the central axis **2056** and passing through both the first and second lateral bores **2074** and **2076**. (Herein, "proximal" and "forward" refer to a direction **2080** that is towards a butt end **2083** of the stock, and "distal" and "rearward" refer to a direction **2084** that is towards a discharge end **2085** of the barrel **2038**.)

The ledge portion **2086** and undercut portion **2087** partially surrounds the base surface **2072** of the bolt face **2054**. The ledge portion **2086** includes an inclined face **88** that faces the base surface **2072** defines a normal vector **2092** (FIG. **85**A) that, during contact with a rim **2148** of a casing **2144** disposed in the recess **2058**, correlates with a retention force exerted thereon. The normal vector **2092** includes an axial component **2094** that is parallel to the central axis **2056** and is directed toward the base surface **2072**. The axial component **2094** of the normal vector **2092** can define an angle θ relative to the normal vector **2092**. In various embodiments, the angle θ is in the range of 25° to 85° inclusive. (Herein, a range that is said to be "inclusive" includes the end point values of the stated range, as well as the values between the end point values.) In one embodiment, the ledge portion **2086** and undercut portion **2087** include a substantially straight portion **2096** that is tangential to the arcuate segment **2060** at a junction point **98**. In one embodiment, the inclined face **2088** of the arcuate segment **2060** is substantially linear in cross-section, to define a frustum shaped profile **2090** (FIG. **85**A).

Alternatively, the ledge portion **2086** can be configured to define other profile shapes. In one embodiment, the ledge portion **2086** includes an arcuate, convex-shaped profile **2090***a* (FIG. **85**B). In this embodiment, a normal vector **2092***a* is defined by the contact line between the rim **2148** of the spent cartridge casing **2174** or cartridge **2140** and the convex-shaped profile **2090***a*. (The rim **2148** and casing **2144** of the spent cartridge casing **2174** or cartridge **2140** is depicted in phantom in FIG. **85**B.) An axial component **2094***a* of the normal vector **2092***a* extends parallel to the central axis **2056**.

The extraction mechanism **2032** also includes a retractable extractor **2100**. In some embodiment, the retractable extractor **2100** is diametrically opposed to the junction point **2098** about the central axis **2056**. In one embodiment, the retractable extractor **2100** is centered at this location. In one embodiment, the retractable extractor **2100** is a claw-type extractor **2102** having a claw portion **2104**, a stem portion **2106**, and a pivot arm portion **2108**. The claw-type extractor **2102** is disposed in the extractor channel **2078** proximate the recess **2058**, with the claw portion **2104** is extendable over the recess **2058** and/or base surface **2072**. The claw portion **2104** can define an apex **2110** at a radially innermost extremity, and a tapered distal face **2112** that slopes distally and away from the apex **2110** with increasing radial distance r from the central axis **2056**.

The apex **2110** may be in axial alignment (with respect to the firearm) with pin **2114**. This minimizes rotation or disengagement of the cartridge rim from the force of the cartridge rim during extraction, enabling the extractor spring to be of minimal force.

The pivot arm portion **2108** of the claw-type extractor **2102** can extend into the first lateral bore **2074** and can be pivotally coupled to a pivot pin **2114** that extends laterally into or through the first lateral bore **2074**. A proximal end **2116** of the of the stem portion **2106** of the claw-type extractor **2102** can extend proximal to the pivot arm portion **2108** and be disposed within the second lateral bore **2076**, with a biasing element **2118** (e.g., a spring) disposed within the second lateral bore **2076**. In one embodiment, the biasing element **2118** exerts a force FB radially outward on the proximal end **2116** of the of the stem portion **2106** of the claw-type extractor **2102**, such that, in a default configuration, the proximal end **2116** of the claw-type retractable extractor **2102** is biased in a rotational position about the pivot pin **2114** that extends the claw portion **2104** of the claw-type retractable extractor **2102** over the recess **2058**.

In one embodiment, the bolt **2052** includes a magazine rail **2120** that is defined on the lower face **2055** of the bolt **2052** and extends substantially parallel to the central axis **2056** along the lower face **2055**. The magazine rail **2120** includes a distal face **2121** that protrudes downward and can be substantially centered about the channel axis **2066**.

The lower face **2055** of the bolt **2052** can further define an ejector channel **20122** within which a stationary ejector **2124** is mounted, the stationary ejector **2124** being stationary relative to the firearm **2030** and including a distal end **2126**. The ejector channel **2122** extends substantially parallel to the central axis **2056** and through the base surface **2072** of the bolt face **2054**. The bolt **2052** can also include a firing pin channel or passage **2128**, within which a firing pin **2132** can be slidingly engaged. The firing pin **2132** includes a distal end **2134** that is selectively extensible into the recess **2058** in a direction normal to the base surface **2072**. In one embodiment, the firing pin **2132** is a rim-type firing pin.

The firing chamber **2042** includes chamber wall **2136** that defines a cylindrical interior chamber **2138** centered about a barrel axis **2139** and having a circular access opening **2142** that faces the breech **2044**, and within which a cartridge **2140** can be mounted and discharged. When mounted in the chamber, the rim **2148** is proximal to the bullet **2143**. The cartridge **2140** is characterized as having the casing **2144** that includes a body or case wall **2146**, a head **2141** having the rim **2148**, and a bullet **2143**. The rim **2148** is further characterized as defining a forward side **2148***a*. The rim **2148** is depicted as being of greater diameter than the case wall **2146**. Standard cartridges of this variety, which are often rimfire cartridges, include the .22 short, the .22 long rifle, and the .22 Winchester Magnum Rimfire (.22 WMR). In certain embodiments, the casing **2144** is of the shouldered variety, having a major diameter **2145** and a minor diameter or neck **2147** joined by a tapered shoulder **2149** (FIG. **85**C). Non-limiting examples of shouldered standard cartridges include the .17 Hornady Magnum Rimfire (.17 HMR) and .17 Winchester Super Magnums (.17 WSM) cartridges. The dimensional specifications for the .17 WSM are also depicted in FIG. **85**C, and presented only as example dimensions of the cartridge **2140**.

Alternatively, the extraction mechanism **2032** can be tailored to extract standard "rimless bottleneck" cartridges with heads that are of approximately the same or smaller diameter as the body for casings where the head projects outward relative to a reduced diameter of the body at the body/rim junction. That is, the head of a rimless bottleneck cartridge does not extend radially beyond the radius of the case wall. Standard cartridges of this variety include, but are not limited to, the .22 Remington and the .17 Remington, which are both centerfire cartridges.

In one embodiment, a ridge **2152** can be formed at a proximal end **2154** of the firing chamber **2042**. The ridge **2152** defines an edge **2156** that is immediately adjacent the circular access opening **2142**, such that when the cartridge **2140** is mounted in the firing chamber **2042**, an exposed portion **2158** of the rim **2148** extends radially outward relative to the edge **2156** of the ridge **2152**. In some embodiments, the edge **2156** of the ridge **2152** is tangential to the circular access opening **2142**.

Referring again to FIGS. **86**A through **86**H, operation of the extraction mechanism **2032** is described in the context of a semi-automatic firearm in an embodiment of the disclosure. In a firing position **2172** (FIGS. **86**A through **86**C), the cartridge **2140** is disposed in the firing chamber **2042** of the firearm **2030**. In the firing position **2172**, in one embodiment, the tapered distal face **2112** of the claw-type extractor **2102** is engaged with the ridge **2152** of the firing chamber **2042**, such that the claw portion **2104** of the claw-type extractor **2102** is pushed radially outward.

The radial outward displacement of the claw portion **2104** causes the claw-type extractor **2102** to rotate about the pivot pin **2114**, such that the proximal end **2116** of the stem **2106** is rotated radially inward against the biasing element **2118**. In this way, the claw-type extractor **2102** retracted, so that the claw portion **2104** is clear of the cartridge **2140** and enabling the rim **2148** of the casing **2144** to be registered against the circular access opening **2142** of the firing chamber **2042**.

In various embodiments, the central axis **2056** of the recess **2058** is parallel to, but not concentric with, the barrel axis **2139**, as best seen in FIG. **86**B. In these embodiments, an outer radius Rr of the rim **2148** at least partially overlaps with the radius R of the arcuate segment **2060** of the ledge portion **2086**, such that when the cartridge **2140** is chambered in the firing position **2172**, the rim **2148** is partially captured by the ledge portion **2086**.

In one embodiment, when in the firing position **2172**, the retractable anchoring bar **2070** extends into an anchoring slot **2171** formed in the breech **2044**, such that a proximal face **2173** of the anchoring bar **2070** registers against a distal face **2175** of the anchoring slot **2171**. In one embodiment, the location and configuration of the anchoring slot **2171** is such that, when the anchoring bar **2070** is registered therein in the firing position **2172**, the bolt face **2054** is in pressing contact with the proximal end **2154** of the firing chamber **2042**.

Upon discharge, a spent cartridge casing **2174** is present in the firing chamber **2042**. For a semi-automatic firearm, the bolt assembly **2046** is disengaged from the firing chamber **2042** by a blowback force FB that also exerts a pressure on the spent cartridge casing **2174** that forces the head **2141** of the casing **2144** against the base surface **2072** of the bolt face **2054**. The blowback force FB causes the bolt assembly **2046** to translate parallel to the central axis **2056** away from the firing chamber **2042**. As the bolt assembly **2046** is translated away from the firing chamber **2042**, the claw portion **2104** of the claw-like extractor **2102** is rotated radially inward, motivated by the biasing element **2118** acting on the proximal end **2116** of the claw-like extractor **2102** (FIG. **86**B). The tapered distal face **2112** of the claw portion **2104** slides on the edge **2156** of the ridge **2152** of the firing chamber **2042**, until the apex **2110** of the claw portion **2104** engages the exposed portion of the rim **2148** of the spent cartridge casing **2174**, thereby hooking the spent cartridge casing **2174**.

As the bolt assembly **2046** is translated in the proximal direction **2080**, the apex **2110** of the claw portion **2104** exerts an axial force FCa against the exposed portion of the rim **2148**, thereby extracting the spent cartridge casing **2174** from the firing chamber **2042** (FIG. **86**C). Initially, the blowback force can continue to exert the blowback force FB and assist in keeping the spent cartridge casing **2174** seated against the base surface **2072** of the bolt face **2054**, as pressure can remain in the firing chamber **2042** during the initial stages of the extraction. The claw portion **2104** also exerts a radial inward force FCr on the spent cartridge casing **2174**. As the spent cartridge casing **2174** is translated out of the firing chamber **2042**, the radial inward force FCr exerted by the apex **2110** of the claw portion **2104** of the extractor can cause the spent cartridge casing **2174** to shift laterally toward the ledge portion **2086**, so that the rim **2148** of the spent cartridge casing **2174** registers against the inclined face **2088** of the ledge portion **2086**. The lateral shifting of the spent cartridge casing **2174** can cause the claw-like extractor **2102** to further rotate about the pivot pin **2114**, which in turn can cause the apex **2110** of the claw portion **2104** to move both radially inward and axially away from the bolt face **2054**.

As the major diameter **2145** of the spent cartridge casing **2174** is extracted in the proximal direction **2080**, the firing chamber **2042**, the interior chamber **2138** of the firing chamber is vented, eliminating the blowback force FB (FIG. **86**D). At this stage of the extraction, the forces exerted on the spent cartridge casing **2174** include the radial inward force FCr exerted at the claw portion **2104**, and a ledge force FL, the ledge force FL having a radial inward component FLr and an axial component FLa, the axial component FLa acting in the proximal direction **2080**. The axial component FLa secures the spent cartridge casing **2174** against the base surface **2072** of the bolt face **2054** as the bolt assembly **2046** is translated within the breech **2044**.

Momentum from the blowback of the discharge continues to translate bolt assembly **2046** parallel to the central axis **2056** in the proximal direction **2080**, with the base surface **2072** of the bolt face **2054** eventually reaching the distal end **2126** of the stationary ejector **2124** (FIG. **86**E) so that the distal end **2126** of the stationary ejector **2124** extends into and/or through the recess **2058**. The protrusion of the stationary ejector **2124** into the recess **2058** projects the spent cartridge casing **2174** distally away from the base surface **2072**. This distal motion causes the rim **2148** of the spent cartridge casing **2174** to slide along the inclined face **2088** of the ledge portion **2086** in the distal direction **2084**, which causes the spent cartridge casing **2174** to move laterally against the claw portion **2104** of the claw-type extractor **2102**. The claw-type extractor **2102** accommodates this lateral movement by rotating radially outward, but maintains contact with the spent cartridge casing because of the bias force exerted on the claw-type extractor **2102** by the biasing element **2118**. As the rim **2148** of the spent cartridge casing **2074** clears the ledge portion **2086**, the rim **2148** initially remains engaged with the apex **2110** of the claw portion **2104**, causing the spent cartridge casing **2174** to pivot about the apex **2110**. The spent cartridge casing then rotates laterally away from the apex **2110** and out of the breach **2044** via an ejection window **2176** (FIG. **86**E).

Referring to FIGS. **87**A and **87**B, certain stages of the extraction are depicted without the aid of a blowback force in an embodiment of the disclosure. Some firearms, such as bolt action or lever action firearms, do not benefit from blowback forces during extraction. Extraction for these devices is provided manually by the user, generally after the firing chamber as fully vented after discharge. The disclosed embodiments are operable without benefit of the blowback force, as depicted in FIGS. **87**A and **87**B.

Additionally, in semiautomatic firearms that do use blowback, at some point the inertia of the bolt assembly moving rearward and the frictional engagement of the casing with the firing chamber wall can overtake the rearward seating force of the cartridge casing, particularly after the pressurization of the firing chamber has dissipated, allowing separation to occur as shown in FIGS. **87**A and **87**B.

For a non-blowback extraction and potentially at a certain point in blowback extraction, the spent cartridge casing **2174** can drag against the chamber wall **2136** of the firing chamber **2042** providing a frictional force FW. The drag FW can cause the spent cartridge casing **2174** to rise off of the base surface **2072** of the bolt face **2054**. The spent cartridge casing **2174** is nevertheless retained within the recess **2058** by the claw portion **2104** of the claw extractor **2102** during the initial stages of the extraction (FIG. **87**A). As the major diameter **2145** proceeds in the proximal direction **2080**, the radial inward force FCr exerted at the claw portion **2104** pushes the rim **2148** towards the ledge portion **86** opposite the claw portion **2104**, and the rim **2148** is captured between the inclined face **2088** and the base surface **2072** (FIG. **87**B). Thus, as the spent cartridge casing **2174** clears the firing chamber **2042**, the spent cartridge casing **2174** is held in equilibrium by the claw portion **2104** and the ledge portion **2086**. The ejection of the spent cartridge casing **2074** then proceeds as described and depicted attendant to FIG. **86**E.

In some instances, the rim **2148** can be canted within the recess **2058** during the extraction, as depicted at FIG. **87**B. The degree to which the rim **2148** is canted depends on several factors, including the uncertainties in the size of the rim **2148** and in the major diameter **2145** introduced by machining tolerances, as well as variability in the frictional drag between the spent cartridge casing **2174** and the firing chamber **2042**. While the precise orientation of spent cartridge casings may vary somewhat during the extraction process, the variability is within a small enough envelope so that the repeatability of the ejection is satisfactory.

Referring to FIGS. **88**A and **88**B, a magazine **2190** is depicted in an embodiment of the disclosure. The magazine **2190** includes a housing **2192** having an upper through-slot **2194** formed thereon. The upper through-slot includes a proximal notch **2196** and a distal notch **2198**. The distal notch **2198** can further define shoulder portions **2202** that lead into the upper slot **2194**. The upper slot **2194** can also define a widened portion **2204** disposed between the proximal and distal notches **2196** and **2198**. A spool **2206** is disposed within the housing **2192**, the spool **2206** rotating about a spindle **2208** (FIG. **89**A) that is supported by the housing **2192**.

In one embodiment, the spool is rotationally biased by a spring **2212** (FIG. **89**A) that is substantially concentric with the spindle **2208**. The spool **2206** includes a plurality of pockets **2214** formed in an outer-most radial surface **2216** of the spool **2206**, each shaped to conform to the casing **2144** of the cartridge **2140**.

In operation, as the spool **2206** rotates about the spindle **2208**, the cartridge **2140** encounters a ramp structure **2218** (depicted in hidden lines) within the housing **2192** that causes the bullet **2143** of the cartridge **2140** to protrude above the housing **2192**, while the rim **2148** remains captured within the housing **2192** in alignment with the proximal notch **2196** of the upper slot **2194** of the magazine **2190** (FIG. **88**B).

Referring to FIGS. **89**A through **89**F, operation of the bolt assembly **2046** and magazine **2190** during resupply the firing chamber **2042** of the firearm **2030** with another cartridge **2140** are depicted in an embodiment of the disclosure. As the bolt **2052** moves forward, the magazine rail **2120** enters the proximal notch **2196** of the upper slot **2194** of the magazine **2190**, so that the distal face **2121** of the magazine rail **2120** makes contact with the rim **2148** of the cartridge **2140** (FIG. **89**A).

The biasing spring **2212** causes the spool **2206** to exert an upward force on the rim **2148**, biasing the rim into the upper slot **2194**, as depicted in FIG. **88**B. As the cartridge **2140** moves in the distal direction **2084**, the rim **2148** becomes aligned with the widened portion **2204**, and pops through the widened portion **2204** due to the force exerted by the biasing spring **2212**. The biasing spring **2212** further causes outermost radial surface **2216** of the spool **2206** to rotate under the rim **2148** of the cartridge **2140**, denoted by rotational arrow **2222** in FIG. **89**B. By this mechanism, the cartridge **2140** is effectively stripped out of the magazine **2190**. The rotation **2222** further elevates the rim **2148**, causing the rim **2148** to enter the channel opening **2068** and to be translated/rotated upward along the channel axis **2066**, sliding along the base surface **2072**. Because the bullet **2143** of the cartridge **2140** is elevated above the upper through-slot **2194** of the magazine **2190**, the cartridge **2140** makes sliding contact with the shoulder portions **2202** of the distal notch **2198** as the cartridge **2140** is thrust forward by the bolt **2052**.

As the cartridge **2140** is translated/rotated along the channel axis **2066**, an outer cylindrical surface **2224** contacts the claw portion **2104** of the claw-type extractor **2102** at an acute angle α relative to an actuation axis **2226** of the claw-type extractor **2102** (FIG. **89**C). The claw-type extractor **2102** is thereby motivated away from the central axis **2056** as the cartridge **2140** slides into place within the recess **2058** (FIG. **89**D).

As the cartridge **2140** continues to be thrust forward, the casing **2144** rides up onto the shoulder portions **2202** of the distal notch **2198** of the magazine **2190**. As the cartridge **2140** is pushed into the cylindrical interior chamber **2138** of the firing chamber **2042**, the outer cylindrical surface **2224** of the casing **2144** comes into sliding contact with the chamber wall **2136**. Because of the close tolerance fit between the casing **2144** and the chamber wall **2136**, the cartridge **2140** becomes righted within the interior chamber **2138** such that the cartridge **2140** is in substantial alignment with the barrel axis **2139** (FIG. **89**E). The alignment causes the rim **2148** of the cartridge **2140** to rotate further upward into the recess **2058** of the bolt **2052**.

The bolt assembly **2046** continues forward until the cartridge **2140** is fully chambered within the firing chamber **2042**. As the bolt face **2054** comes into pressing contact with the proximal end **2154** of the firing chamber **2042**, the anchoring bar **2170** extends into the anchoring slot **2171** to secure the bolt **2052** against the firing chamber **2042** (FIG. **89**F). The firearm **2030** is thereby in the firing configuration **2172** of FIGS. **86**A through **86**C, with the rim **2148** captured by and in contact with the inclined face **2088** of the ledge portion **2086**, as depicted in FIG. **86**C.

When used herein, the terminology "connect to" or "attach to" do not require direct component to component connection and intermediate components may be present.

All of the features disclosed in this specification (including the references incorporated by reference, including any accompanying claims, abstract and drawings), and/or all of the steps of any method or process so disclosed, may be combined in any combination, except combinations where at least some of such features and/or steps are mutually exclusive.

Each feature disclosed in this specification (including references incorporated by reference, any accompanying claims, abstract and drawings) may be replaced by alternative features serving the same, equivalent or similar purpose, unless expressly stated otherwise. Thus, unless expressly stated otherwise, each feature disclosed is one example only of a generic series of equivalent or similar features.

The invention is not restricted to the details of the foregoing embodiment (s). The invention extends to any novel one, or any novel combination, of the features disclosed in this specification (including any incorporated by reference references, any accompanying claims, abstract and drawings), or to any novel one, or any novel combination, of the steps of any method or process so disclosed The above references in all sections of this application are herein incorporated by references in their entirety for all purposes.

Although specific examples have been illustrated and described herein, it will be appreciated by those of ordinary skill in the art that any arrangement calculated to achieve the same purpose could be substituted for the specific examples shown. This application is intended to cover adaptations or variations of the present subject matter. Therefore, it is intended that the invention be defined by the attached claims and their legal equivalents, as well as the following illustrative aspects. The above described aspects embodiments of the invention are merely descriptive of its principles and are not to be considered limiting. Further modifications of the invention herein disclosed will occur to those skilled in the respective arts and all such modifications are deemed to be within the scope of the invention.

We claim:

1. A semiautomatic firearm chambered for rimfire cartridges, the semiautomatic firearm comprising: a receiver connecting to a barrel extending forwardly from the receiver, the barrel with a firing chamber, a bolt assembly movable along a central axis within the receiver and slidingly engaged therewith, the bolt assembly comprising: a bolt body with a firing pin extending through the bolt body, the bolt assembly slidingly movable into and out of an in-battery position; a movable member constrained by, extendable outwardly from, and retractable inwardly to the bolt body, the movable member having an distal end engaged with a cam surface external to the bolt assembly, the cam surface fixed with respect to the receiver, the cam surface including a transition portion whereby when the bolt assembly moves forwardly and rearwardly with respect to the receiver, the transition portion moves the movable member between an extended position and a retracted position, the bolt assembly further comprising a carrier positioned in the bolt body and having a ramp portion with a ramp surface engaged with the movable member at a surface on the movable member away from the distal end of the movable member, the carrier movable forwardly and rearwardly within the bolt body, the bolt assembly further comprising a manual handle extending from the carrier outwardly from an ejection port defined in the receiver.

2. The semiautomatic firearm of claim **1**, wherein the cam surface is a surface on the receiver.

3. The semiautomatic firearm of claim **1**, wherein the barrel is chambered for one of .17 Winchester Super Magnum cartridge and .17 Hornady Magnum Rimfire.

4. The semiautomatic firearm of claim **1**, wherein the movable member is pivotal with respect to the bolt body, a pivot axis extending transverse to a barrel axis.

5. The semiautomatic firearm of claim **4**, wherein when the bolt assembly is in an in battery position and the manual handle is pulled rearwardly, the carrier moves rearwardly in the bolt body and the movable member is retractable inwardly to the bolt body.

6. The semiautomatic firearm of claim **2**, wherein the manual handle connects to the carrier through a handle intermediary member.

7. The semiautomatic firearm of claim **2**, wherein the carrier is biased forwardly with respect to the bolt body whereby the ramp portion biases the movable member toward an extended position.

8. A semiautomaticfirearm configured as a rifle, the semiautomaticfirearm comprising:a receiver connecting to a barrel extendingforwardlyfrom the receiver, the barrel with a firing chamber, a bolt assembly movable along a central axis within the receiverand slidingly engaged therewith, the bolt assembly comprising: a bolt body with a firing pin extendingthrough the bolt body, the bolt assembly slidingly movable into and out of an in-battery position; a movable member constrained by, extendable outwardly from, and retractable inwardlyto the bolt body, the movable member having an distal end engaged with a cam surface external to the bolt assembly, the cam surface fixed with respect to the receiver, the cam surface including a transition portion whereby when the bolt assembly moves forwardly and rearwardly with respect to the receiver, the transition portion moves the movable member between an extended position and a retracted position, the bolt assembly further comprising a carrier positioned in the bolt body, the carrier having a ramp portion with a ramp surface engaged with the movable member at a surface on the movable member, the carrier movable forwardly and rearwardly within the bolt body, wherein when the carrier is forward in the bolt body the movable member is extending outwardly; and wherein a manual handle connects to the carrier and extends outwardly from an ejection port.

9. The semiautomatic firearm of claim **8**, wherein the carrier is exposed at the ejection port of the firearm.

10. The semiautomatic firearm of claim **8**, wherein the manual handle connects to the spanning memberthrough a handle intermediary member.

11. The semiautomatic firearm of claim **8**, wherein the movable member is pivotal with respect to the bolt body about an axis that extends transverse to axis of the firearm barrel.

12. The semiautomatic firearm of claim **8**, wherein the carrier spans a width of the bolt body and is exposed on both lateral sides of the bolt body.

13. The semiautomatic firearm of claim **11**, wherein when the movable member extends outwardly from the bolt body, it extends upwardly from the bolt body.

14. The semiautomatic firearm of claim **8**, wherein when the movable member is retracted inwardly the firing pin is impeded from moving forward.

15. The semiautomatic firearm of claim **8**, wherein when the bolt assembly is in an in battery position and the manual handle is pulled rearwardly, the carrier moves rearwardly in the bolt body and the movable member retracts inwardly to the bolt body.

16. The semiautomatic firearm of claim **15**, wherein the carrier is biased forwardly with respect to the bolt body thereby biasing the movable member toward an extended position.

17. A semiautomatic firearm chambered for rimfire cartridges and configured as a rifle, the semiautomatic firearm comprising: a receiver connecting to a barrel extending forwardly from the receiver, the barrel with a firing chamber, a bolt assembly movable along a central axis within the receiver and slidingly engaged therewith, the bolt assembly comprising: a bolt body with a firing pin extending through the bolt body, the bolt assembly slidingly movable into and out of an in-battery position; a movable member pivotally constrained by, extendable outwardly from, and retractable inwardly with respect to the bolt body, the movable member having an distal end engaging with a cam surface external to the bolt assembly, the cam surface fixed with respect to the receiver, the cam surface including a transition portion whereby when the bolt assembly moves forwardly and rearwardly with respect to the receiver, the transition portion moves the movable member between an extended position and a retracted position, the bolt assembly further comprising a carrier positioned in the bolt body and having a ramp portion with a ramp surface engaged with the movable member at a surface on the movable member away from the distal end of the movable member, the carrier movable forwardly and rearwardly within the bolt body, the bolt assembly further comprising a manual handle extending from the carrier outwardly from an ejection port defined in the receiver.

18. The semiautomatic firearm of claim **17**, wherein when the bolt assembly is in an in battery position and the manual handle is pulled rearwardly, the carrier moves rearwardly in the bolt body and the movable member is retractable inwardly to the bolt body.

19. The semiautomatic firearm of claim **18**, wherein when the movable member is retracted inwardly the firing pin is impeded from moving forward.

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