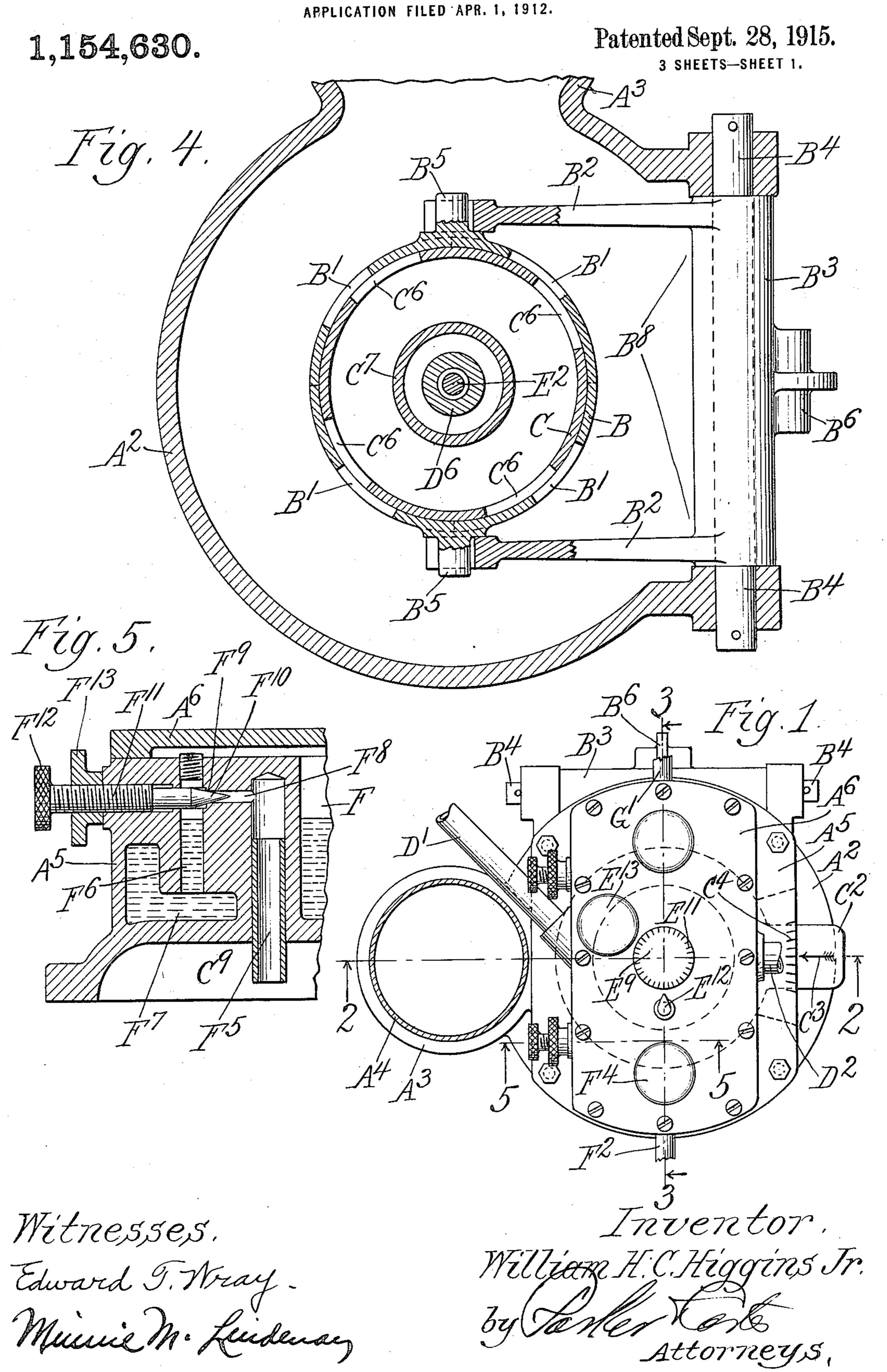
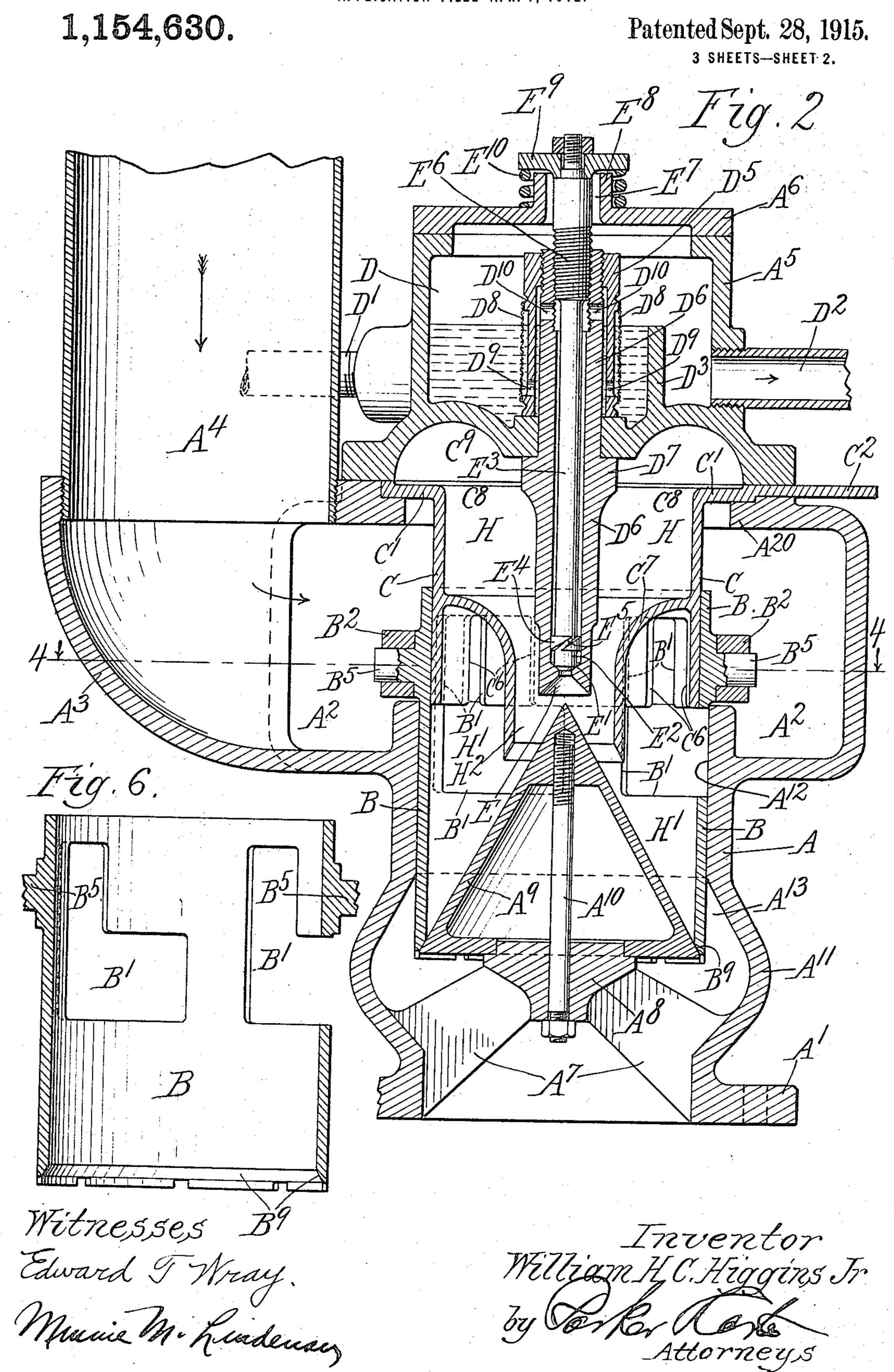
W. H. C. HIGGINS, JR. CARBURETER.



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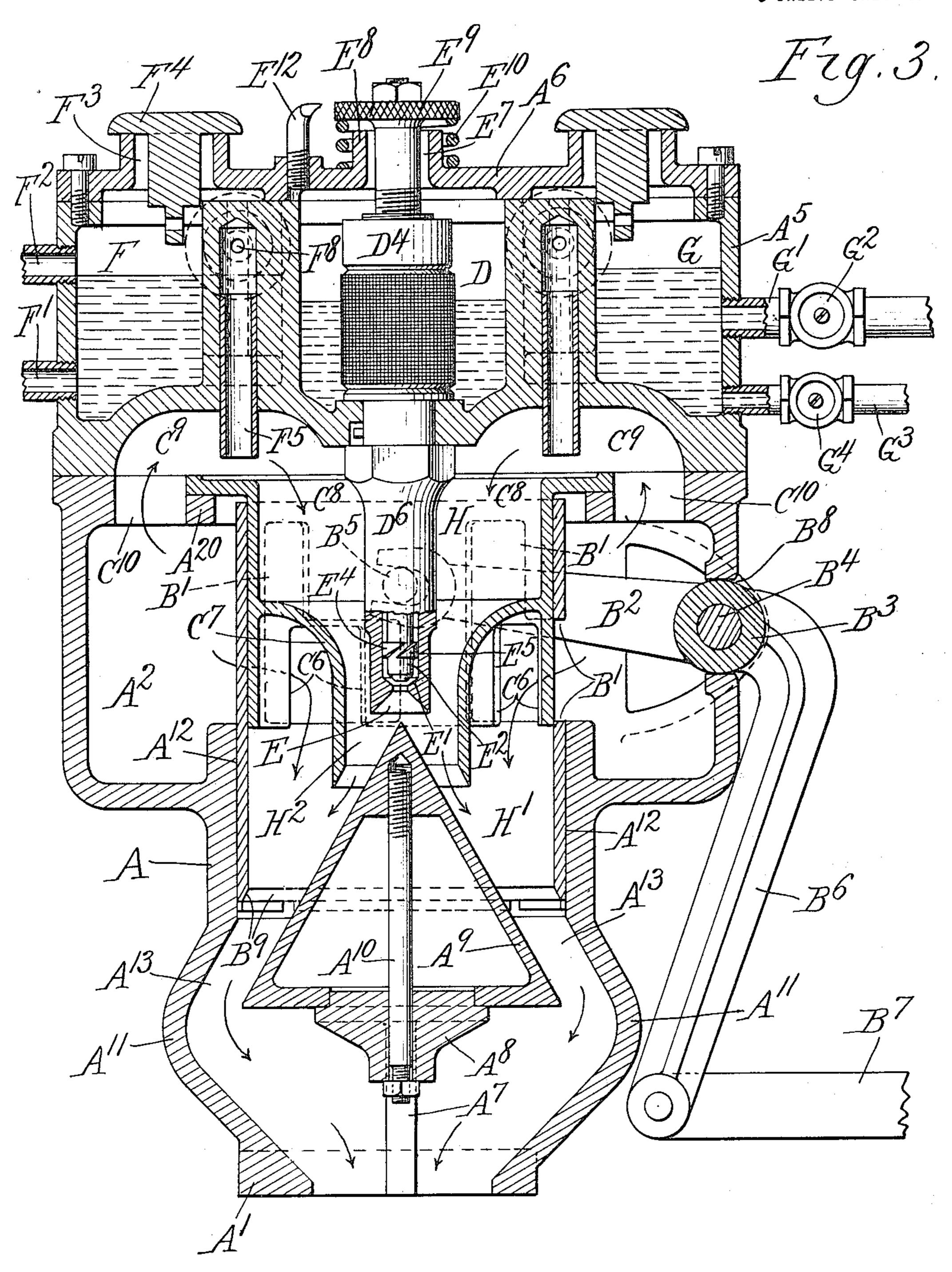
APPLICATION FILED APR. 1, 1912.



W. H. C. HIGGINS, Jr. CARBURETER. APPLICATION FILED APR. 1, 1912.

1,154,630.

Patented Sept. 28, 1915.
3 SHEETS—SHEET 3.



Witnesses. Edward T. Wray. Munic M. Lindenson. Inventor.
William H.C. Higgins Ir.
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UNITED STATES PATENT OFFICE.

WILLIAM H. C. HIGGINS, JR., OF LAPORTE, INDIANA, ASSIGNOR TO M. RUMELY COMPANY, OF LAPORTE, INDIANA, A CORPORATION OF INDIANA.

CARBURETER.

1,154,630.

Specification of Letters Patent.

Patented Sept. 28, 1915.

Application filed April 1, 1912. Serial No. 687,722.

To all whom it may concern:

Be it known that I, William H. C. Higgins, Jr., a citizen of the United States, residing at Laporte, in the county of Laporte and State of Indiana, have invented a certain new and useful Improvement in Carbureters, of which the following is a specification.

My invention relates to improvements in carbureters and is illustrated diagrammatically in one form in the accompanying

drawing, wherein-

Figure 1 is a plan view; Fig. 2 is a section along the line 2—2 of Fig. 1; Fig. 3 is a section along the line 3—3 of Fig. 1; Fig. 4 is a section along the line 4—4 of Fig. 2; Fig. 5 is a section along the line 5—5 of Fig. 1; Fig. 6 is a detail elevation in part section of the sliding sleeve.

Like parts are indicated by like letters

throughout the several figures.

The housing A is provided at its bottom with the flange A1 whereby it may be attached to the intake manifold, contains the 25 annular air passage or chamber A² from which projects the elbow A³ to engage the air pipe A4, and carries on its upper side the reservoir containing housing A5, having the cover A⁶. The spider arm structure A⁷ 30 which projects inwardly from the housing A immediately above the flange A¹ carries the hub A⁸ on which is mounted the upwardly extending conical deflector A⁹ held in position by the stud A¹⁰. The casing A 35 is enlarged as indicated at A¹¹ about the base of the deflector A⁹ and contains the vertically disposed cylindrical bearing surface A¹² located above, concentric with, and slightly removed from the deflector A⁹.

40 The sleeve B which is slidably mounted in the bearing A¹² is provided with the L shaped ports B1 which have their larger portion below, and is actuated by the forked levers B2, which project from the sleeve B3 45 on the shaft B4 and engage the lugs B5 projecting from the sleeve. The lever B6 which projects outwardly from the sleeve B³ is actuated by the link B7 which operates in response to the engine or to any any other suitable controlling means. The port B⁸ in the outer wall of the chamber A² is substantially closed by the sleeve B3, although it is not necessary that there be in any sense an air-tight joint here. The coni-55 cal seat B⁹ at the bottom of the sleeve B is

in opposition to and parallel with the surface of the deflector A⁹, and the sleeve when in the lower position rests upon this deflector to close the annular mixture discharge port A¹³ between the deflector A¹⁰ 60 and the bottom edge of the bearing surface A¹²

face A¹².

The sleeve C which is rotatably mounted in the sleeve B, which sleeve is itself slidably mounted thereon, is provided at its up- 65 per end with the annular horizontally extended flange C¹ rotatably mounted in the casing A, is held in position by the casing A⁵ and is provided with the operating arm or lever C² which projects above the cham- 70 ber A² and carries the indicator C³ in opposition to the scale C⁴ on the housing A⁵. The sleeve C extends downwardly only as far as the upper edge of the bearing A¹² and is provided with the ports C⁶ in register 75 with the ports B¹ and contains the inwardly and downwardly extending curved deflector C⁷ located above and to one side of the ports C⁶.

The sleeves B and C thus form a two 80 part mixing and carbureting chamber C⁸, which is provided with the mixture discharge port A¹³ as above indicated, the controllable and adjustable auxiliary air intake ports B¹, and which has an open upper end or port C⁹ into which discharge the fixed and unvarying air intake passages C¹⁰ which extend across the flange A²⁰ and communicate with the annular chamber A².

The housing A⁵ contains the fuel reser- 90 voir D which is provided with the oil supply pipe D¹ and the overflow pipe D². The partition or overflow wall D3 located in the chamber D adjacent the overflow pipe D² limits the height of the liquid therein. The 95 fuel valve D4 is centrally located within the reservoir D and discharges downwardly therefrom into the vacuum chamber at a point immediately above the deflector A⁹. The valve D4 is held in position in the bot- 100 tom of the reservoir D by the sleeve D⁵ which is screw threaded at its upper end on the upper end of the nipple D6, is out of contact therewith except at the screwthreaded connection and draws the shoulder 105 D' on the nipple firmly against the bottom of the casing A⁵ to hold the valve nozzle in position. The sleeve D⁵ is surrounded toward its base with the wire screen or netting D⁸ and is provided with the passages 110

D⁹ guarded by the netting D⁸ and leading from the chamber D adjacent its base to the space between the nipple D⁶ and the sleeve D⁵. The nipple D⁶ is provided with the 5 passages D¹⁰ located adjacent its top and leading from the space between the nipple and the sleeve to the interior of the nipple.

The nipple D⁶ itself is provided at its lower end with the downwardly extending 10 conical discharge nozzle E, above which is the conical valve seat E¹. The valve E² in opposition to the valve seat E1 is carried by the valve stem E³, which stem is guided at its lower end in the tubular interior of the 15 nipple by the flange or shoulder E4 having the feed slots E⁵ therein. The stem E³ is provided at its upper end with the enlarged screw threaded portion E⁶ in screw threaded engagement with the upper end of the 20 nipple. The cover A⁶ is provided with the aperture E⁷ surrounded by the upwardly extending flange E⁸ through which passes the screw threaded portion E⁶. The thumb screw E⁹, which presses against the spring 25 E¹⁰ and is provided with the index marks E¹¹ in opposition to the pointer E¹², is provided for the purpose of adjusting the position of the valve to control the fuel supply.

The interior of the reservoir D may be in-30 spected through the port or passage covered by the cap E¹³. The housing A⁵ contains on one side of the reservoir D the water reservoir F provided with the supply pipe F¹ and the overflow pipe F² and the man-35 hole F³ covered by the cap F⁴. The discharge nozzle or tube F⁵ discharges downwardly into the passage C9. The passage F6, which communicates at its base by means of the passage F^7 with the interior reservoir F, 40 communicates at its upper end with the tube F⁵ by means of the cross passage F⁸ containing at one end the conical valve seat F⁹ in opposition to which is the needle valve F¹⁰ carried by the screw threaded valve stem 45 F¹¹ which may be controlled by the thumb screw F¹² and held in position by the lock nut F^{13} .

The gasolene or priming fluid reservoir G on the opposed side of the housing A⁵ 50 may be filled with gasolene or priming fluid through the pipe G¹ controlled by the valve G² and may be drained through the ripe G³ controlled by the valve G4. This reservoir discharges by means of a valve and nozzle, 55 exactly the same as the water valve and nozzle, into the passage C⁹ above the mixing chamber. The two part mixing and carbureting chamber C⁸ is divided by the curved deflector C⁷ into the upper and lower cham-60 bers H, H¹ connected by the fixed more or less cylindrical passage H2 through the deflector in which is located the fuel nozzle. The chamber H¹ is a true valve controlled vacuum chamber. It is provided with the 65 constant fixed air inlet port H2, the variable

opening and closing mixture discharge port A¹³, and the variable opening and closing auxiliary air intake port C6, which latter ports may also be adjusted by the rotating sleeve so that the mixture discharge and 70 auxiliary air intake ports respond unequally to the movement of the sliding sleeve. This movement of the sliding sleeve together with the position of the rotary adjusting sleeve controls the vacuum in this chamber, and it 75 is this vacuum which controls the flow of air from the chamber H¹. The chamber H¹ operatively includes the passages C⁹ leading from the annular passage A2, and therefore the water and gasolene or priming supplies 80 and the fuel supply are all located in this mixing vacuum chamber H¹. I have called this chamber H¹ a vacuum chamber, because it is positively and absolutely controlled by the vacuum, although there is not neces- 85 sarily any controllable vacuum in the chamber itself, because this chamber is not provided with any adjustable or controllable ports. It is of course evident that under some circumstances and for some fuels and oper- 90 ating conditions, the deflector C⁷ may be dispensed with without in any way changing the operation of the device, since then there would be but a single vacuum chamber in which the vacuum would be controlled by 95 the adjustable air intake, the fixed air intake ports and the relation which their total effective area at one time bore to the total effective area of the mixture discharge port. For the sake of convenience, however, and under 100 some circumstances, for the purpose of protecting the oil nozzle from cross currents which might be set up throughout the auxiliary air intake ports to discharge against it, I have provided the deflector C⁷ which cuts 105 the vacuum chamber into two parts.

It will be evident that while I have shown in my drawings an operative device, still many changes might be made in the particular mechanical arrangement and structure 110 without departing materially from the spirit of my invention, since many forms of carbureter having different mechanical arrangement and movement might be produced which would operate exactly as does 115 this device.

The use and operation of my invention are as follows: As the engine operates it draws its adjustable fuel charge from and through the carbureter, the suction of the engine 120 causing the air to flow through the carbureter, carrying with it the combustible fluids. The air enters the carbureter, passes down through the upwardly projecting air tube into the annular air chamber surround- 125 ing the vacuum or carbureting chamber proper. Here the air is divided, part of it going up through the top of the rotatable sleeve and thus down into the vacuum chamber, another portion going in through the 123

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walls of the vacuum chamber through the registering ports in the sleeves and thus into the vacuum chamber. Here the air currents again unite and pass down across the conical 5 deflector through the mixture discharge port and the enlarged portion of the carbureter and down into the intake manifold not shown. The air which passes through the fixed passages and ports surrounds the lower 10 end of the water nozzle and is supplied with a suitable amount of water. It then goes on down into the vacuum chamber, passing the fuel nozzle, the vacuum in the chamber, or the induced suction of the air current as the 15 case may be, drawing the fuel out of the nozzle, and it is downwardly discharged in a finely atomized condition into the mixing chamber. It then accompanies the air down along the conical deflector, being mixed with the free air which enters the auxiliary controllable air intake ports and passes out in a finely atomized condition to the engine.

The vacuum in the carbureting chamber draws the fuel positively down the fuel noz-25 zle. The fuel as it leaves the reservoir first enters the base of the outer sleeve of the cylindrical valve, is drawn up to the top of the sleeve and at that point enters the interior of the nipple. It then passes down 30 the nipple to the nozzle. It will be noted that the screw threaded connections at the top of the nipple and between the nipple and the sleeve and between the nipple and the needle valve will be more or less loose 15 and open and therefore the siphon cannot be marked, although the air which will enter at these points will not be of sufficient quantity to prevent the drawing out of the fuel by the positive suction caused by the static 40 vacuum of the vacuum in the mixing chamber. The same is true of the priming and water supplies. While the tubes or passages through which these liquids pass, discharge into the carbureter at a point below the bot-45 tom of the reservoir and might at first glance seem to act as a siphon, still that action is absent, since the parts are carefully made so as not to be absolutely air tight. The suction is sufficient to draw fuel up and 50 out, but the weight of the fuel itself, acting as a siphon, is not. The fuel and water supplies, it will be noted, discharge downwardly and the passage to the air which carries the fuel and water is also in a down-55 ward direction, thus as the velocity of the air current varies in response to the demands of the engine and pulsates with each stroke, there will never be any tendency for the finely divided particles of water and fuel 60 to assume a velocity in the direction opposed to the direction in which they must be carried by the air column to reach the engine, and during the intervals between the intake strokes of the engine when the air 65 column is substantially at rest, all that will

happen will be that the finely divided particles will pass downwardly under the action of gravity in the direction in which they ought normally to go. The deflector arranged about the fuel nozzle and above the 70 auxiliary air intake ports, separates them from the nozzle and thus prevents their too freely upsetting and reducing the static vacuum which must be found about the nozzle in order to draw out the fuel with the re- 75

quired accuracy.

The function of the conical deflector at the base of the vacuum chamber is to gradually deflect the air currents in order to prevent the presence of eddy currents and in so order to prevent any violent change in the direction of motion of the fuel-laden air, since owing to the varying specific gravities of water, oil and air, such eddy currents and such violent changes in direction would 35 cause the air to throw down the water and oil, owing to their greater density, and thus a poorer mixture would be discharged to the engine. This is important, in view of the fact that this carbureter is designed pri- 90 marily for use of the heavier grades of fuel oils which are not vaporized at ordinary varying temperatures, but which are merely atomized in the vacuum chamber and carried from the carbureter to the engine as of finely divided mist borne upon the air current.

The sliding sleeve or valve, which may as indicated be controlled from the engine or from any other suitable means, when raised, 100 opens the mixture discharge port and when. lowered closes it, thus shutting off the supply of air and fuel. It will be further noted that when the sleeve is raised, the ports therein are brought into register with the 105 ports in the rotatable sleeve, and thus the auxiliary air intake ports are open. The amount of their opening, however, is controlled by the position of the rotatable sleeve. This sleeve is rotated by means of 110 the lever projecting outwardly therefrom, and set at such a position that for any position of the sliding sleeve a suitable mixture

will be produced.

When the engine is running light, the 115 sliding or throttling sleeve will be lowered to almost close the mixture discharge port. This will operate to very materially decrease the auxiliary air intake opening and therefore most of the air which enters the car- 120 bureting chamber will come in through the fixed air intake port. This port, however, will be made so small that the auxiliary air intake ports must be slightly open to permit sufficient air to enter the carbureter. 125 This auxiliary air intake port will be adjusted until the vacuum in the carbureter in response to the demands of the engine will feed a comparatively rich mixture to the carbureter, because when but small quanti- 130

ties of mixture are admitted to the engine, the depression will decrease and therefore a richer mixture is needed to ignite. As the demands of the engine increase, the recip-5 rocating sliding throttling sleeve will be raised, thus opening the mixture discharge port and permitting an increased amount of mixture to reach the engine. This will increase the compression in the engine and 10 therefore a leaner mixture may be used. Therefore the proportion of fuel with respect to the amount of air may be decreased, but the amount of air being increased, the fuel supply also will have to be increased, 15 but not so rapidly as the air supply, and this may be done by increasing the vacuum.

When once the rotatable adjusting sleeve has been positioned in any certain engine, the movement of the slidable throttling 20 sleeve will control the actual effective openings of the adjustable air intake and mixture discharge ports both with respect to each other and with respect to the fixed passage through the deflector so as to produce 25 in response to the suction of the engine a suitable vacuum in the lower vacuum chamber. This vacuum will control the inflow of air, oil and water to and through the upper vacuum or mixing chamber, so as to give a suitable and satisfactory combustible mixture. Under some circumstances, and preferably, there will be a positive static vacuum in the upper chamber and this static vacuum will operate to suck out of the oil 35 and water supply nozzles a suitable amount of liquid, which when mixed with the air entering the fixed port, and also in the lower vacuum chamber the air entering the adjustable port, will give a suitable propor-40 tion of oil and water so that a combustible charge will be fed to the engine cylinder. It will be evident, however, that such a static vacuum need not necessarily be had in the upper chamber, and in fact, under some cir-45 cumstances and conditions, the static vacuum in the upper chamber might be negligible, and it is conceivable that the velocity of the air alone might be relied upon to draw out a suitable quantity of oil and water. In any event, however, it is the vacuum which controls, for if there is a vacuum in the entire two part chamber, it of liquid. If, however, there is only a vacu-⁵⁵ um in the lower chamber, this vacuum will so adjust and control the air current that still a suitable quantity of liquid will be fed, and I have called the whole chamber a vacuum chamber, because in any event, whether vacuum prevails throughout with the same intensity, or not, it is a vacuum controlled chamber and the fuel and water feeds are controlled by the vacuum and by the vacuum alone, whether indirectly or di-

rectly.

I claim:

1. A carbureter comprising a cylindrical mixing chamber, air intake and mixture discharge ports surrounding either end thereof, an auxiliary air intake port inter- 70 mediate them, and a conical deflector closing one end of said chamber and forming one side of the mixture discharge port.

2. A carbureter comprising a mixing chamber made up of a plurality of cylin- 75 ders mounted one upon the other, one of them slidable, the other rotatable, registering ports in said cylinders controlled by their respective movements, and a mixture discharge port controlled only by said slid- 80

able cylinder.

3. A carbureter comprising a vacuum chamber made up of a plurality of movable cylinders mounted one upon the other, one rotatable, the other slidable each with re- 85 spect to the other, a fixed air intake port at one end of one of said cylinders, a mixture discharge port at the opposed end of the other cylinder, auxiliary registering intake ports in the walls of said cylinders.

4. A carbureter comprising a cylindrical mixing chamber, an annular air supply passage surrounding the upper end thereof, fixed air intake ports leading from said passage and discharging axially into said 95 chamber, auxiliary air intake ports leading from said passage and discharging radially into said chamber, a mixture discharge port at the other end of the chamber, and a water supply located above said fixed air 100

intake port.

5. A carbureter comprising a cylindrical mixing chamber, an annular air supply passage surrounding the upper end thereof, fixed air intake ports leading from said pas- 105 sage and discharging axially into said chamber, auxiliary air intake ports leading from said passage and discharging radially into said chamber, a mixture discharge port at the other end of the chamber, and a prim- 110 ing fluid supply located above said fixed air intake port.

6. A carbureter comprising a two part vacuum chamber, a fixed intake port connected with one part of said chamber, a 115 fixed discharge leading from said part to the other part of said chamber, adjustable will operate to draw out a suitable amount auxiliary air intake and mixture discharge ports in the other part of said vacuum chamber, a fuel supply discharging into the 120 first part of said vacuum chamber adjacent the port discharging to the second part of the chamber and a water supply connecting with the first part of said chamber adjacent the intake port.

> 7. A carbureter comprising a two part vacuum chamber, a fixed intake port connected with one part of said chamber, a fixed discharge leading from said part to the other part of said chamber, adjustable 180

auxiliary air intake and mixture discharge ports in the other part of said vacuum chamber, a fuel supply discharging into the first part of said vacuum chamber adjacent 3 the port discharging to the second part of the chamber, a water supply connecting with the first part of said chamber adjacent the intake port and means for controlling said mixture discharge and said auxiliary

air intake port in unison together with a 1) separate adjusting means for said auxiliary port.

Signed at Laporte, Indiana, this 19th day of March, 1912.

WILLIAM H. C. HIGGINS, JR.

Witnesses:

MELVIN A. CHASE, JOHN N. HAYN.