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**Binder et al.**

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(54) **DEVICE AND METHOD FOR SHAPING WIRE ENDS IN A CIRCUMFERENTIAL DIRECTION**

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See application file for complete search history.

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*Primary Examiner* — David P Bryant

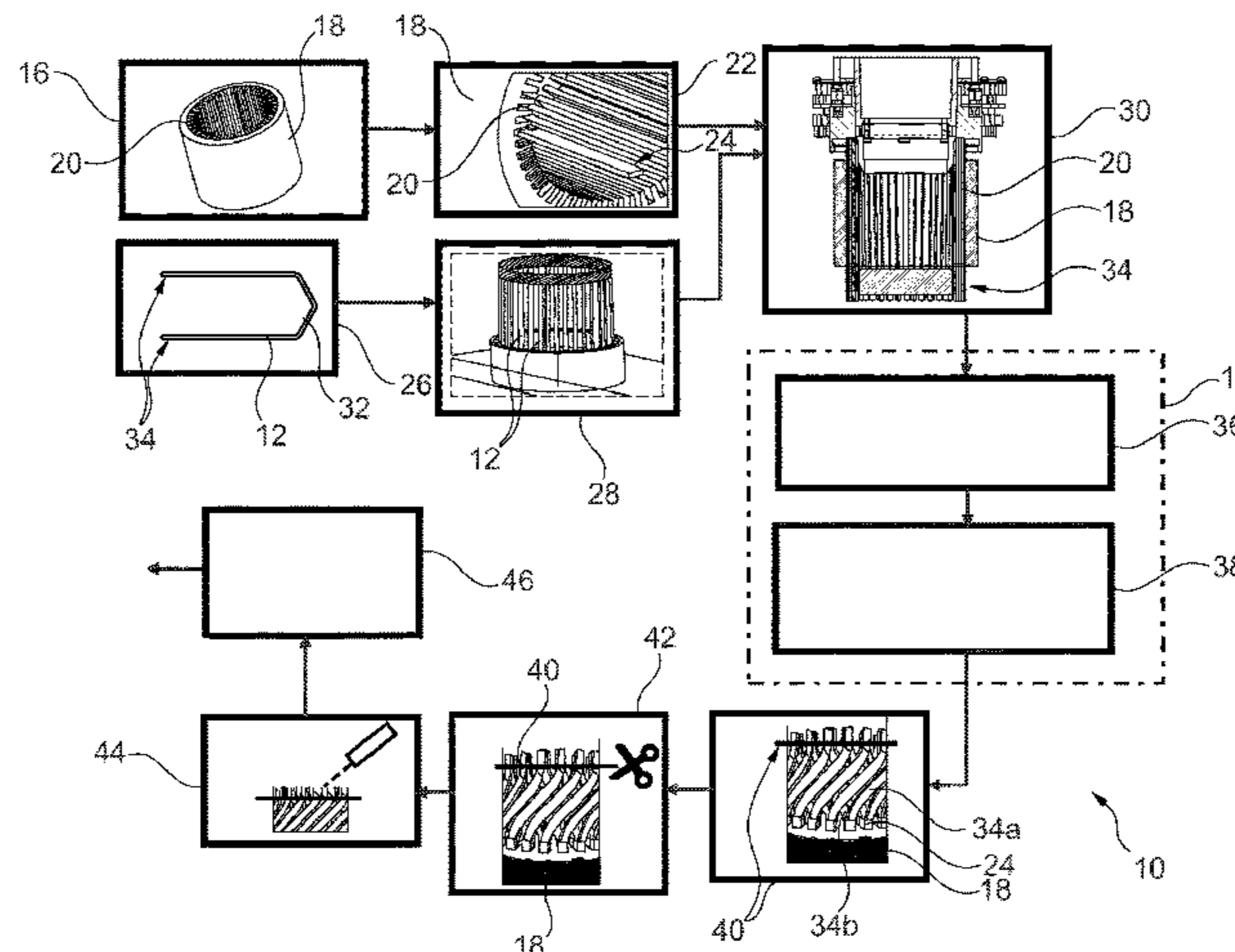
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(57) **ABSTRACT**

To automate a sub-process during the production of a component, which is to be equipped with coils, of an electric machine, a wire end shaping device is used, which is equipped with coils, of an electric machine, for shaping wire ends which protrude from an annular housing of the component, comprising a bending device for bending wire ends in a circumferential direction, a relative movement device for moving the housing and the bending device in a relative manner in an axial direction, and a controller. The receiving and rotating units, which receive the wire ends, of the bending device are held to be axially static relative to one another, wherein differences in length compensation and/or in a turning angle of the wire ends are handled by means of different movement profiles of the rotational movements of the receiving and rotating units.

**8 Claims, 24 Drawing Sheets**



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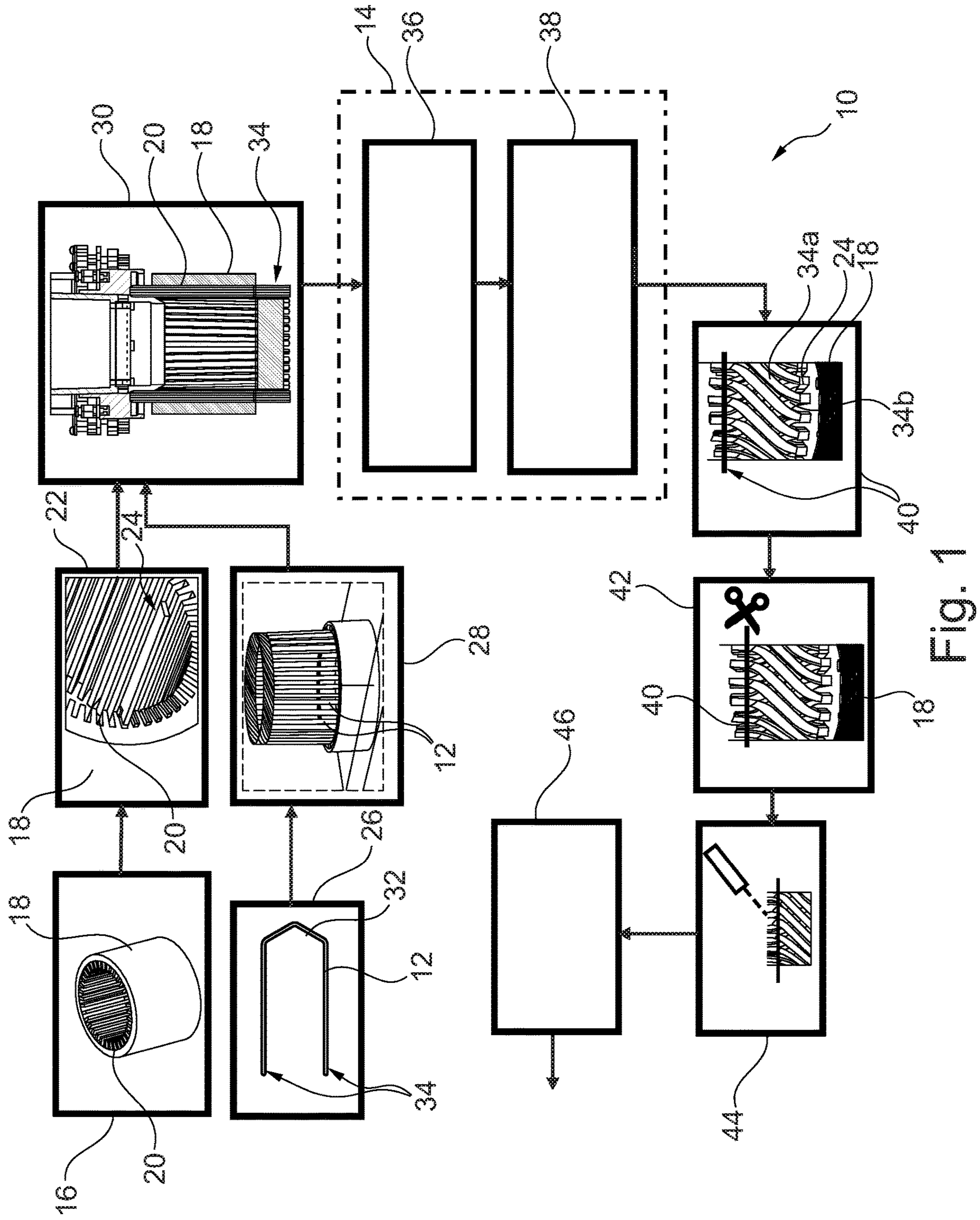
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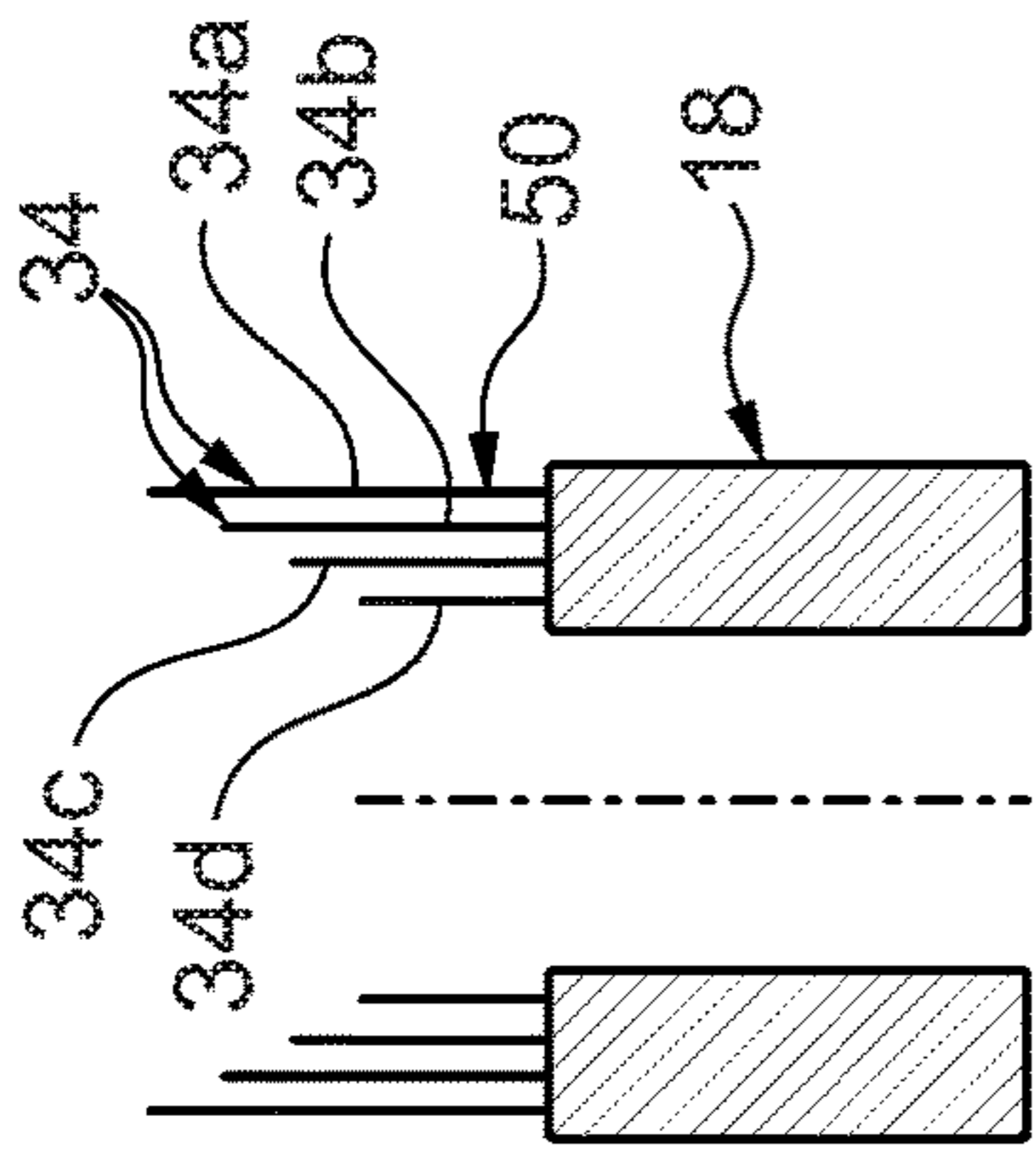


Fig. 2

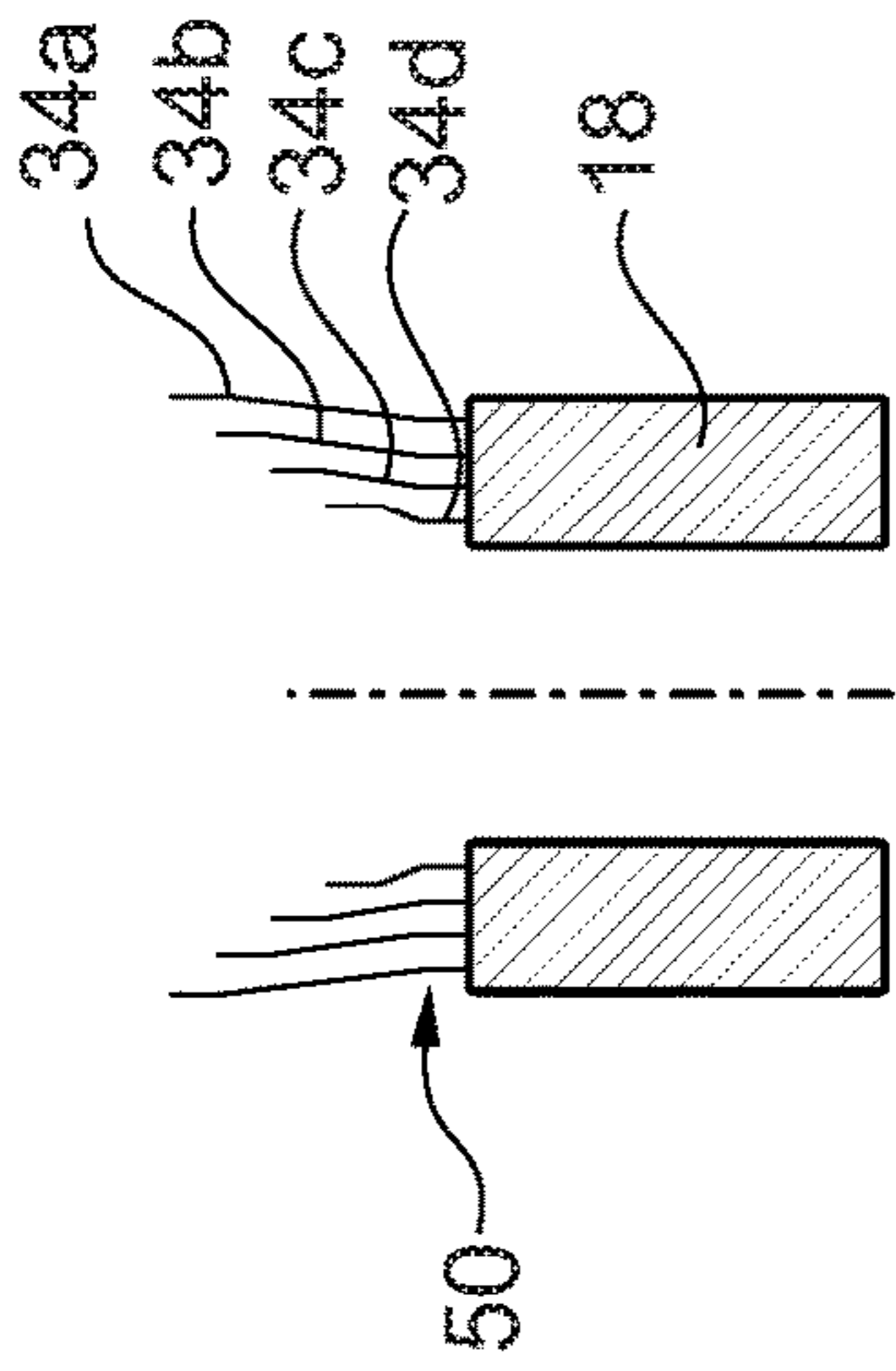


Fig. 3

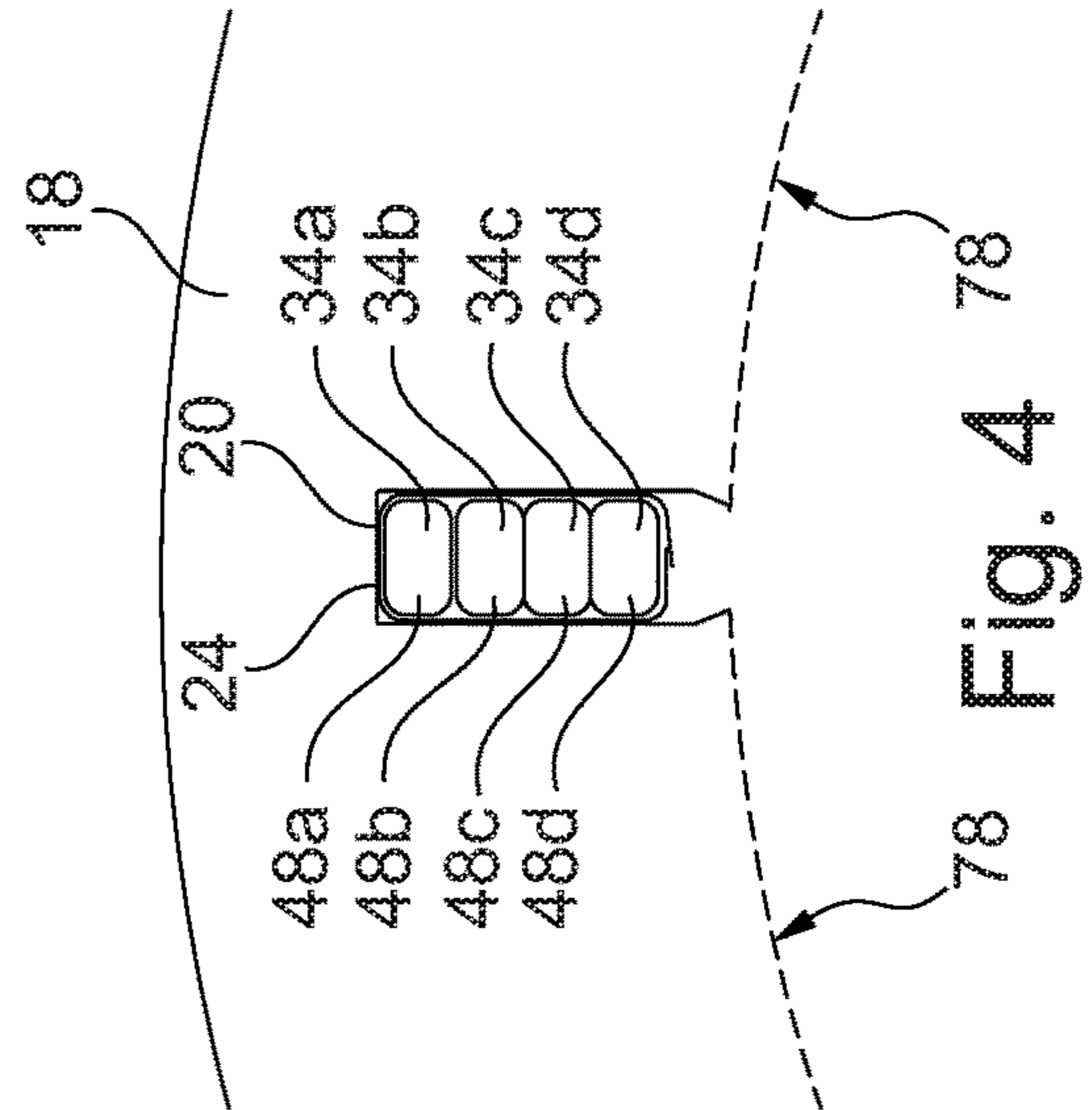


Fig. 4

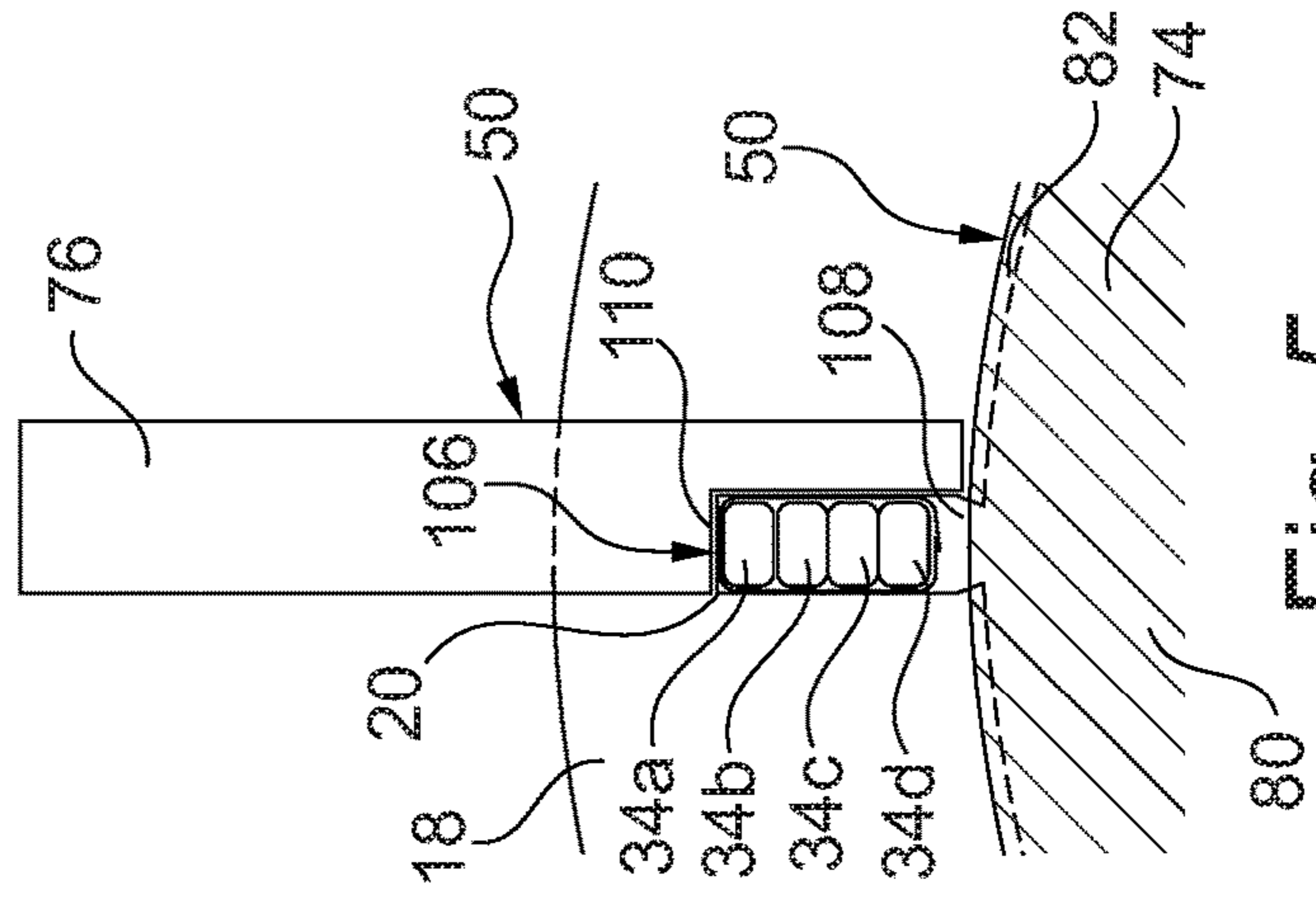


Fig. 5

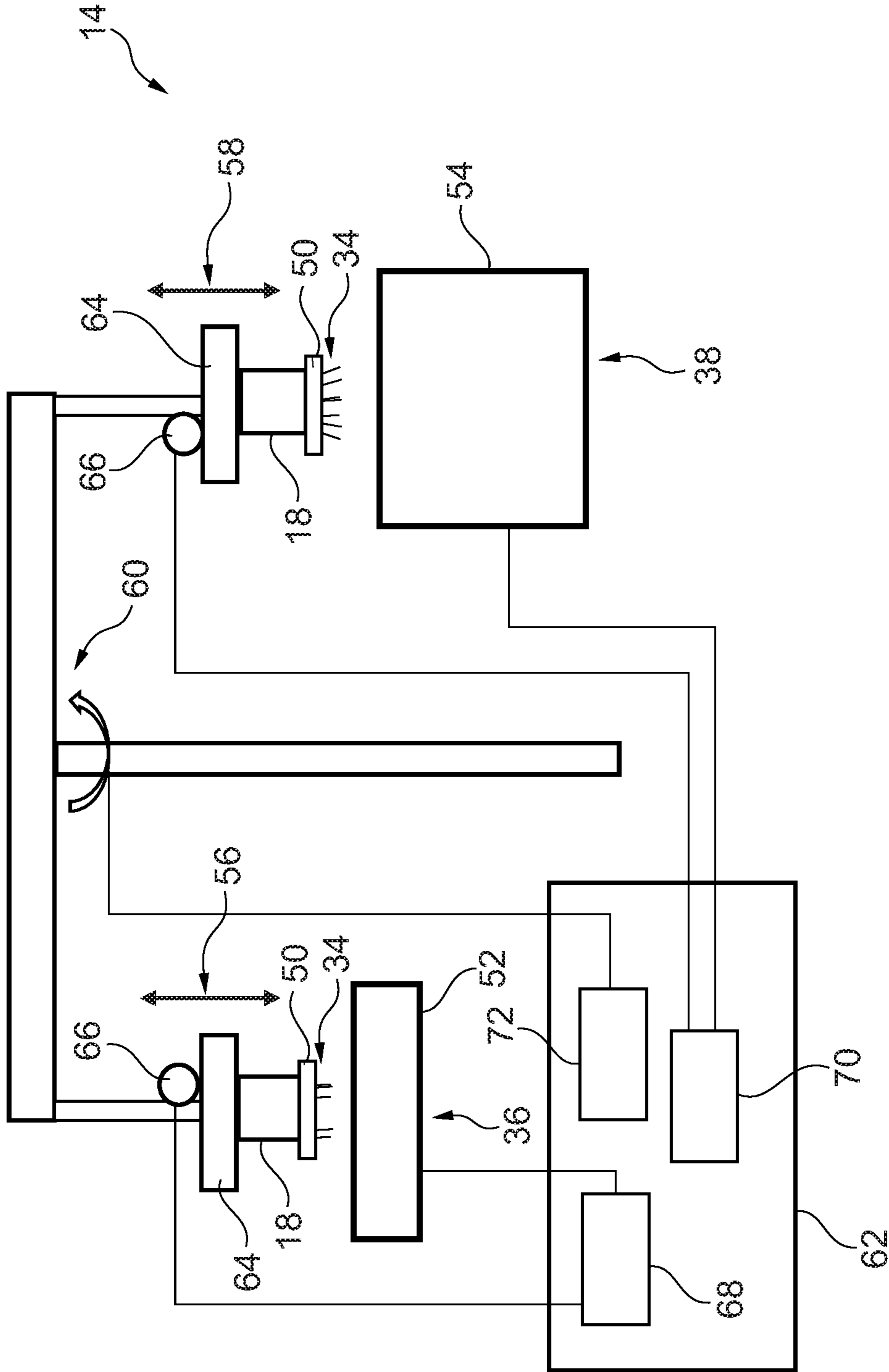


Fig. 6

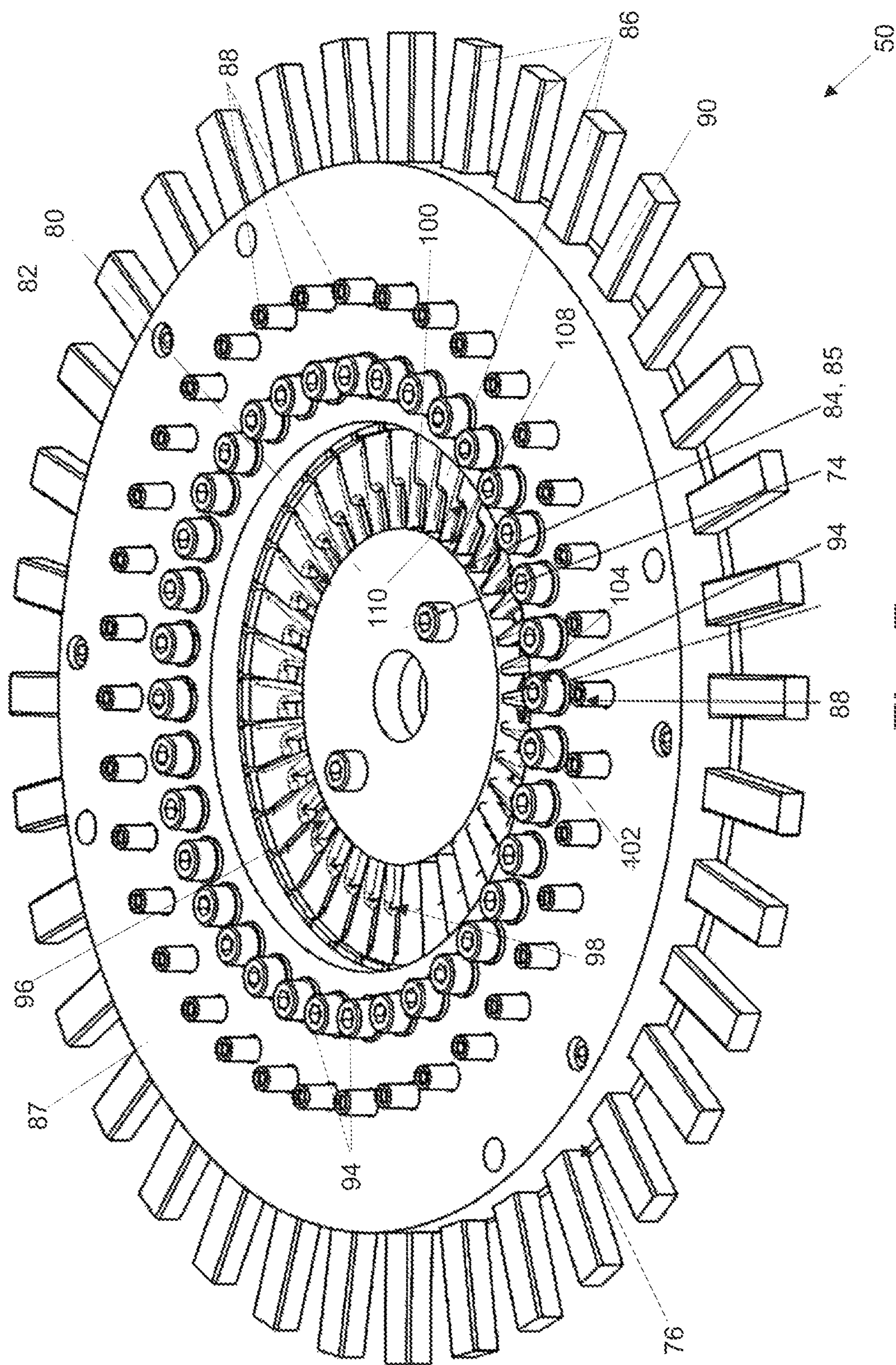


Fig. 7



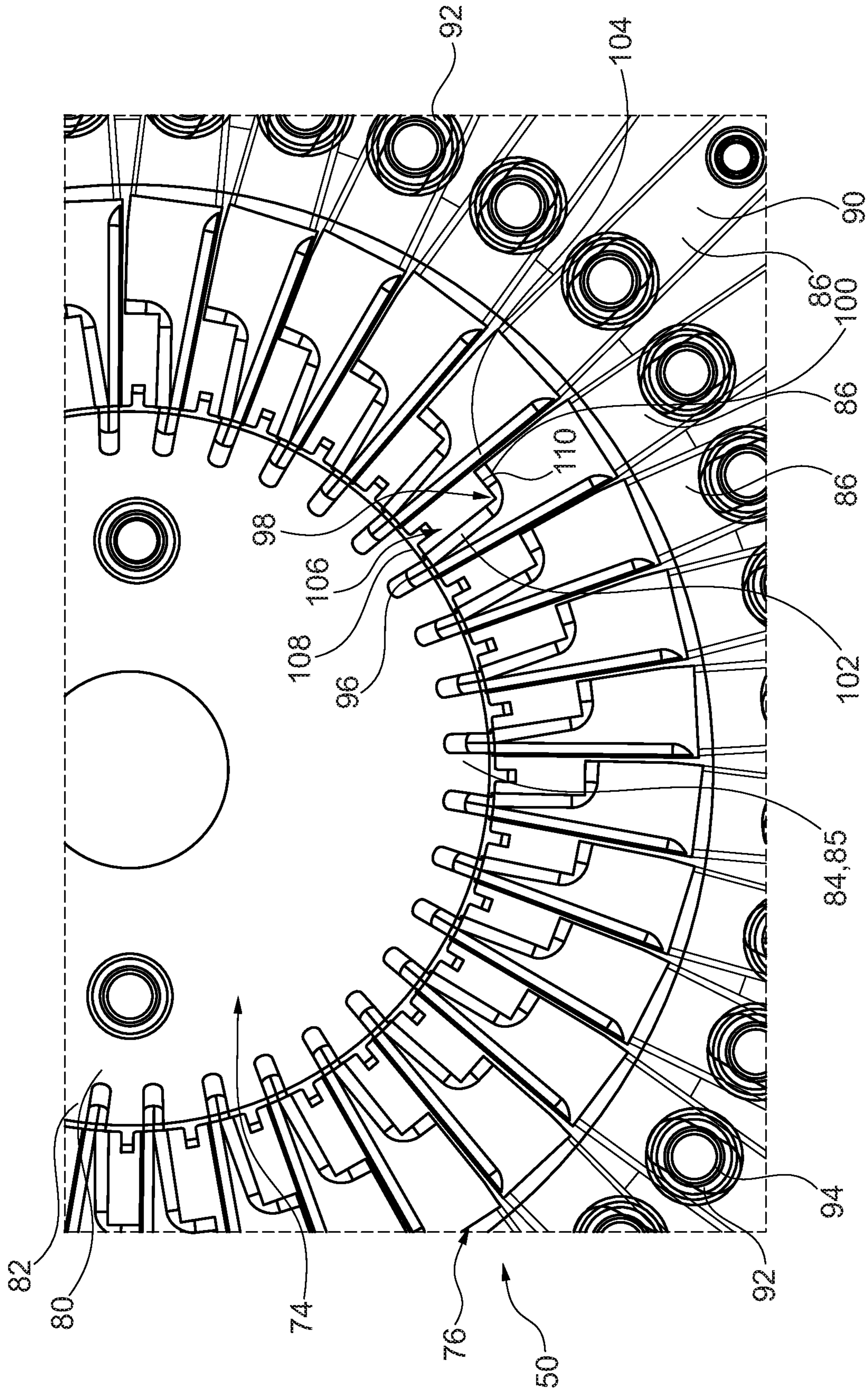


Fig. 8

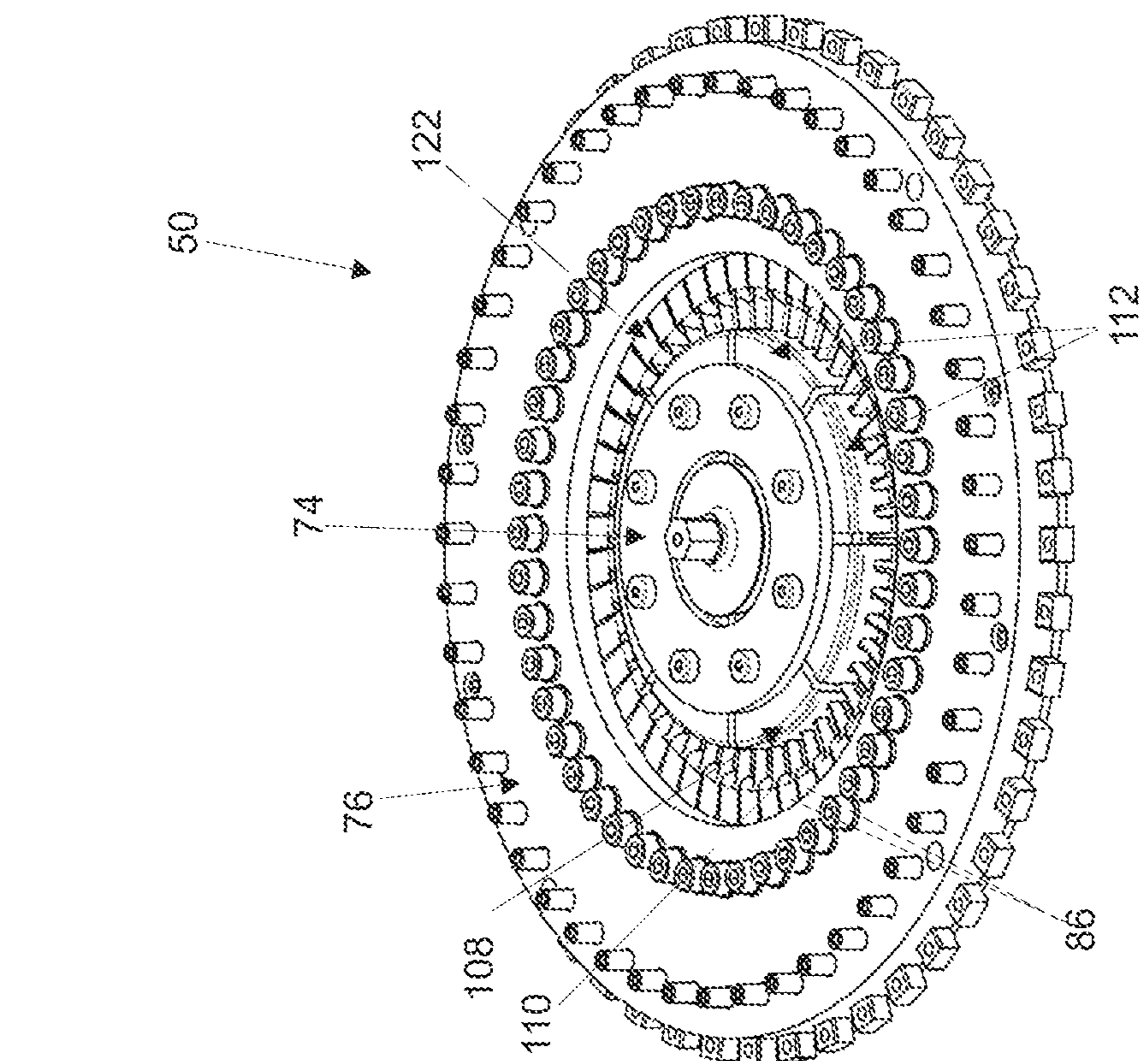


Fig. 10

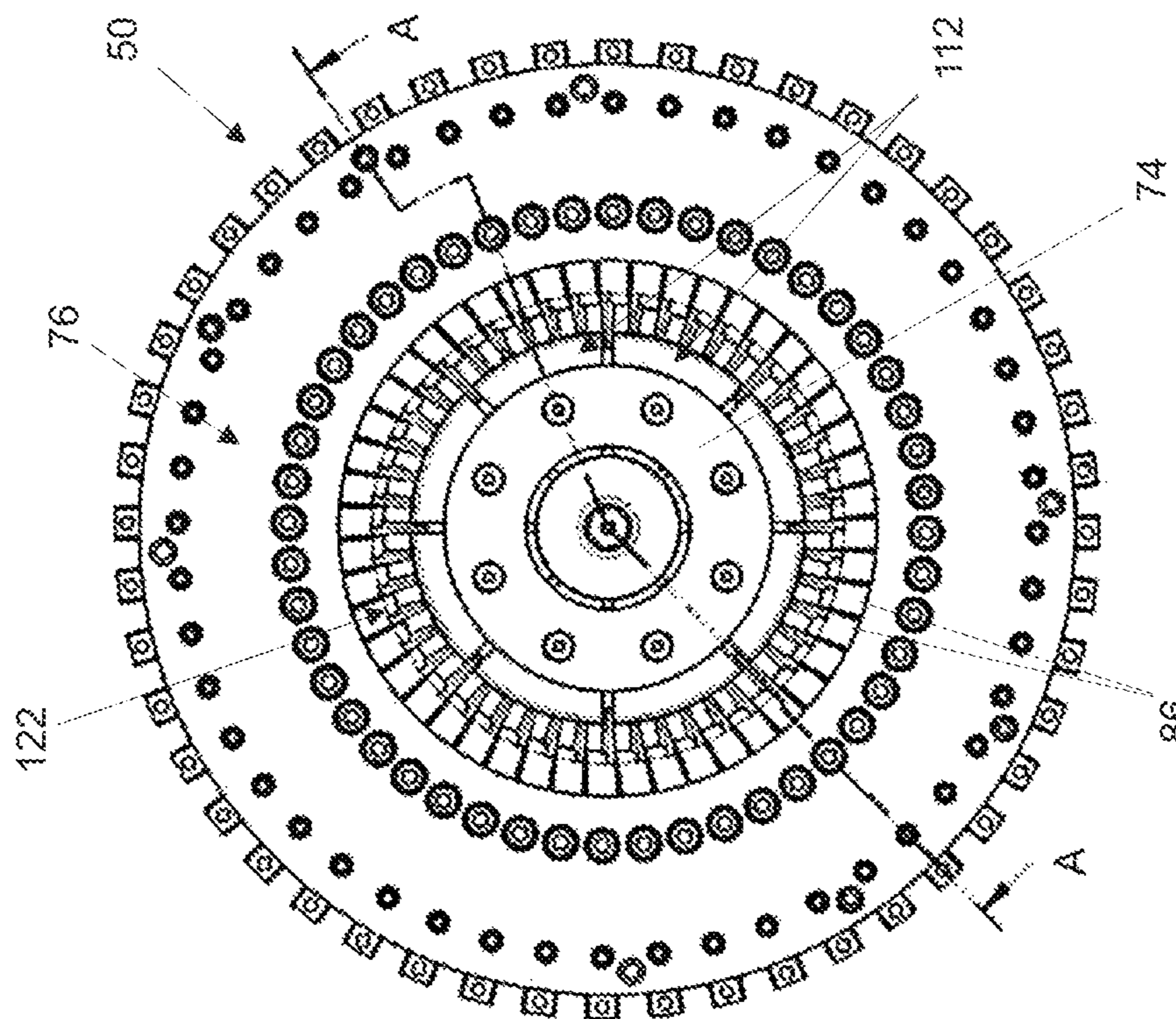


Fig. 9



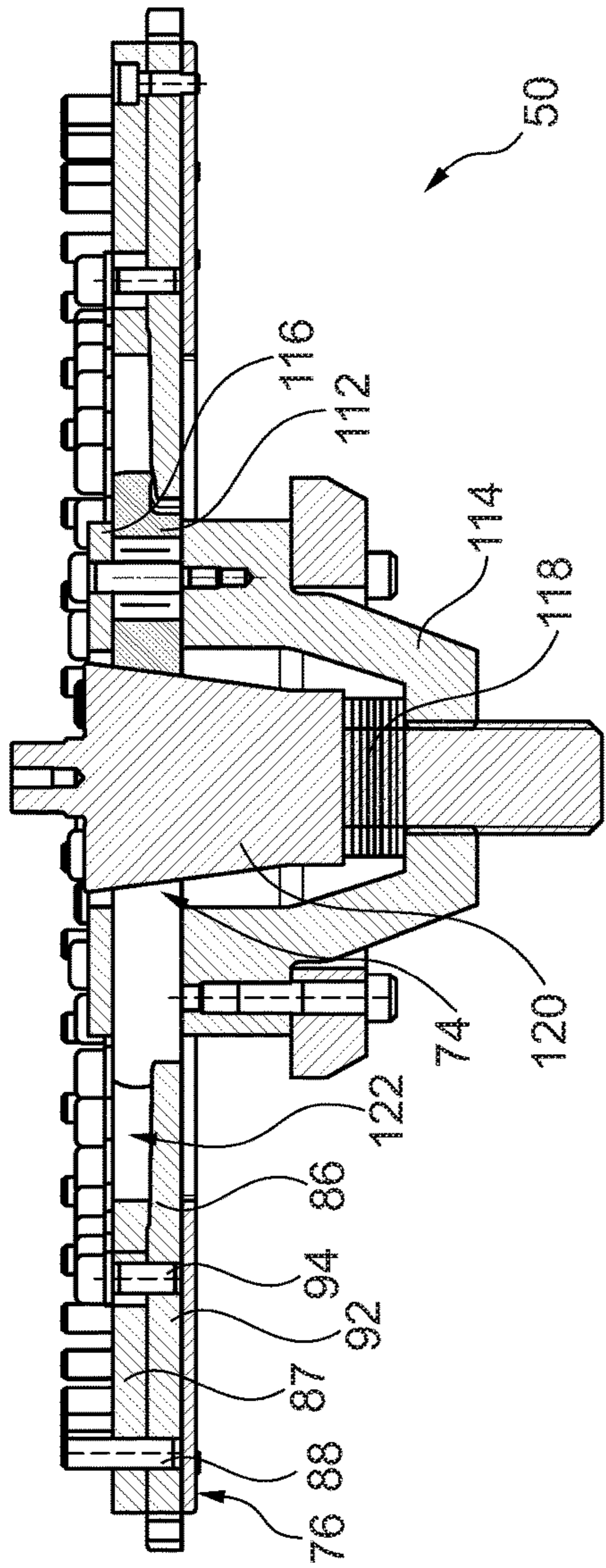


Fig. 11

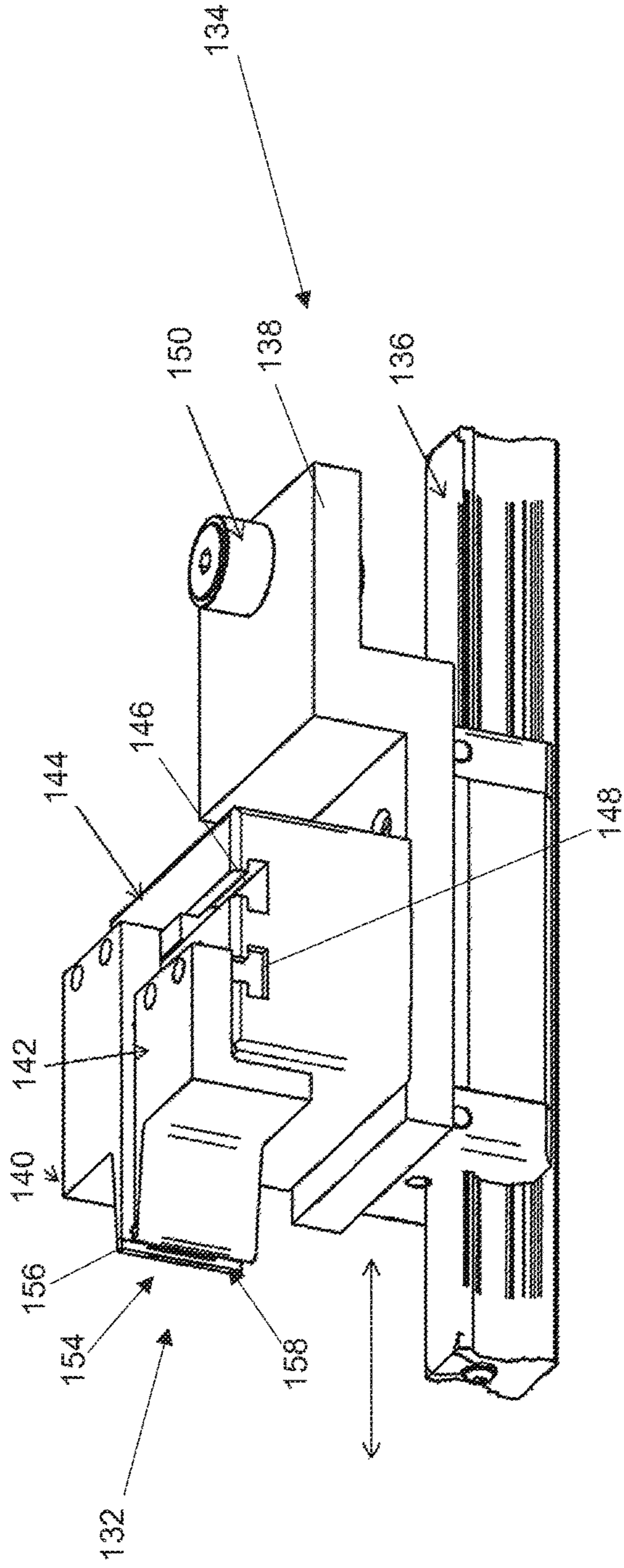


Fig. 14

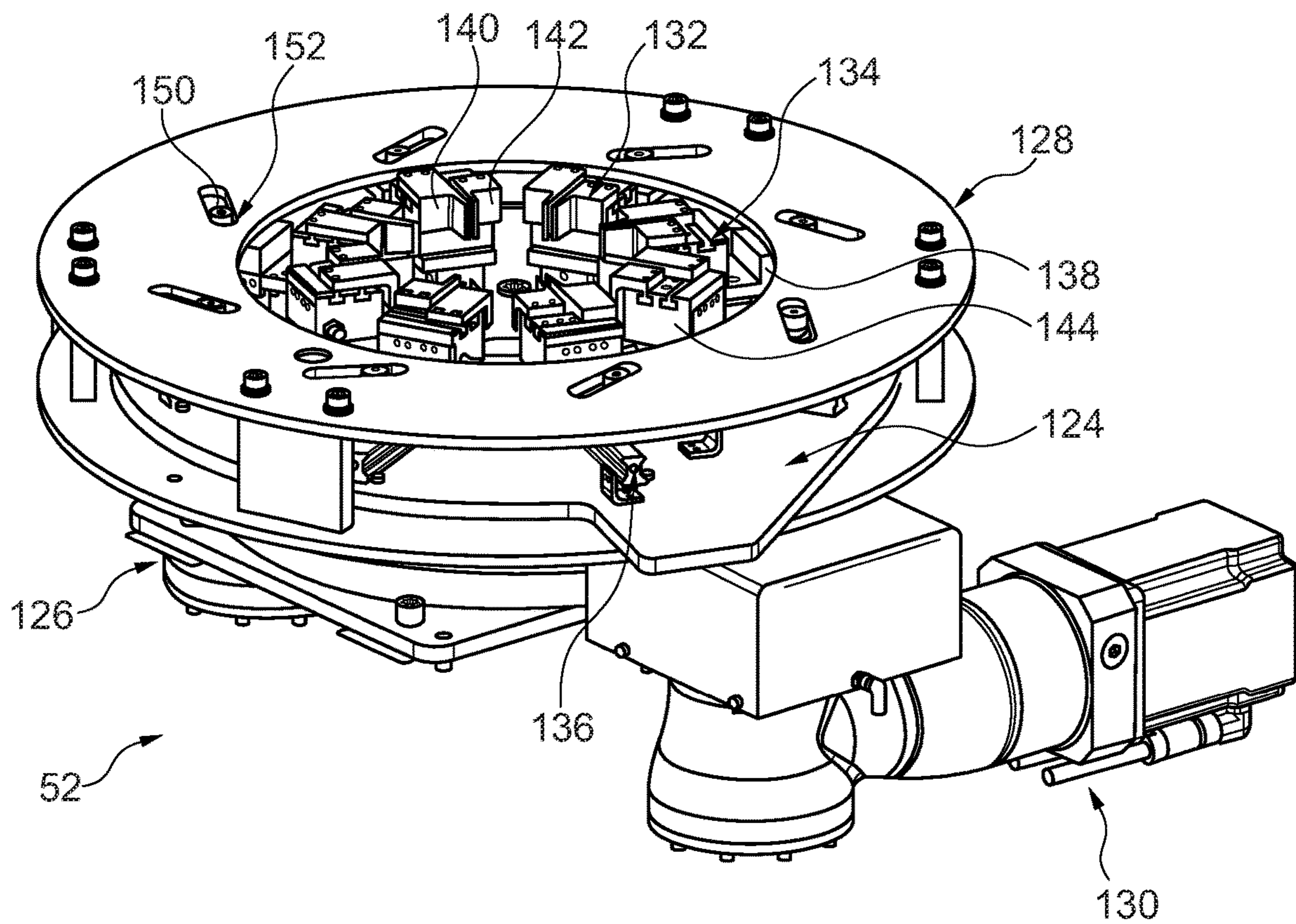


Fig. 12



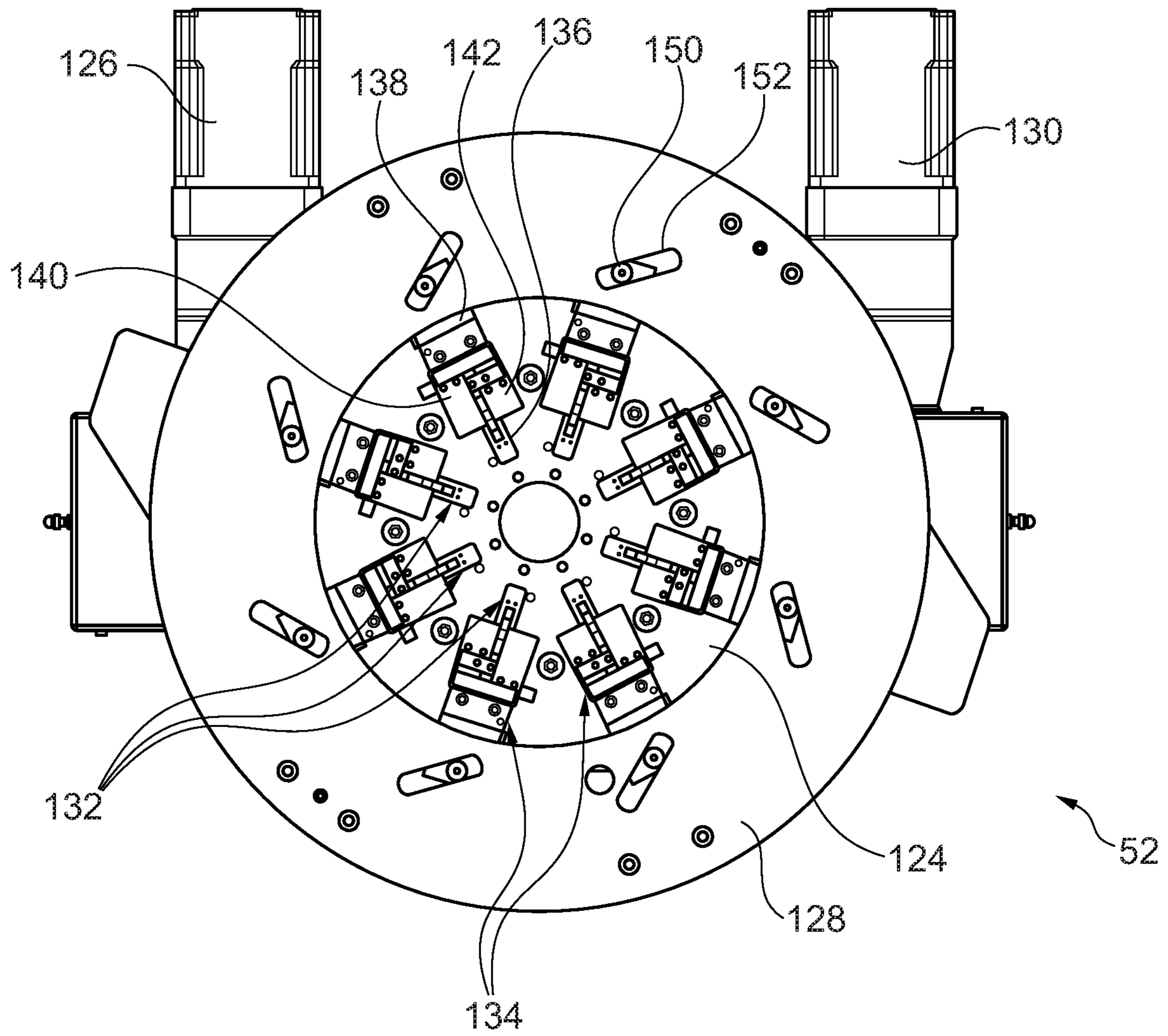


Fig. 13

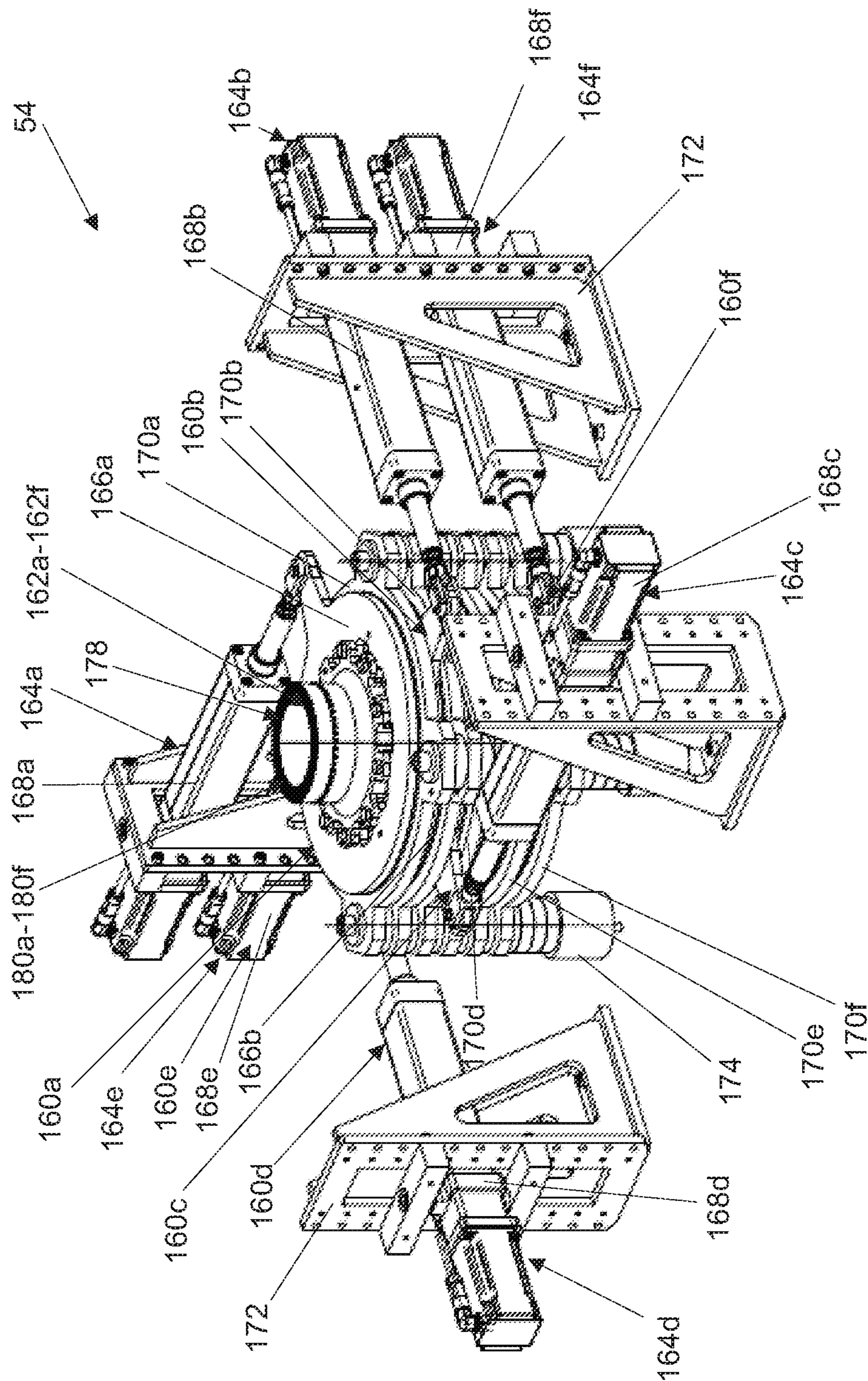


Fig. 15



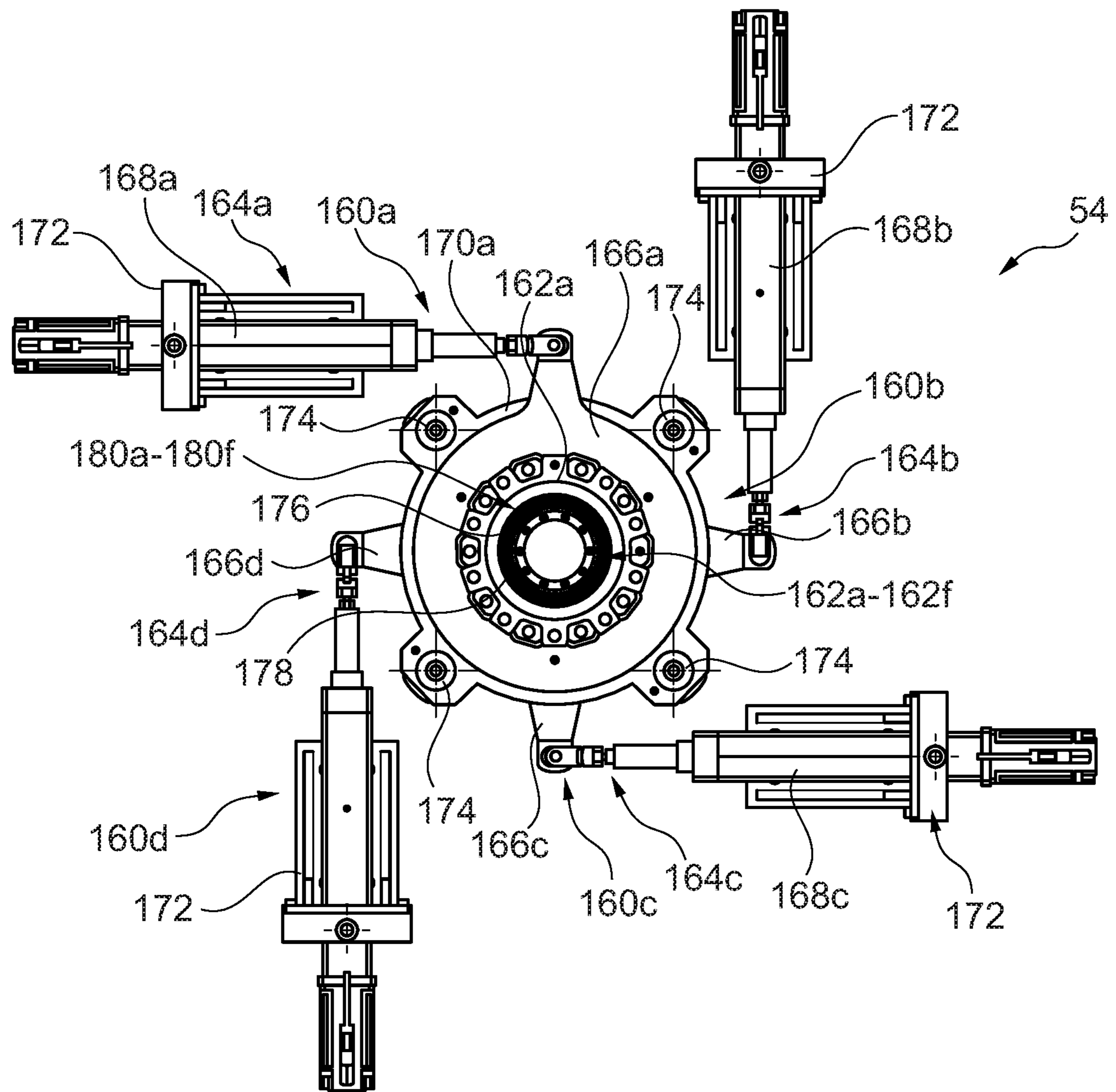


Fig. 16

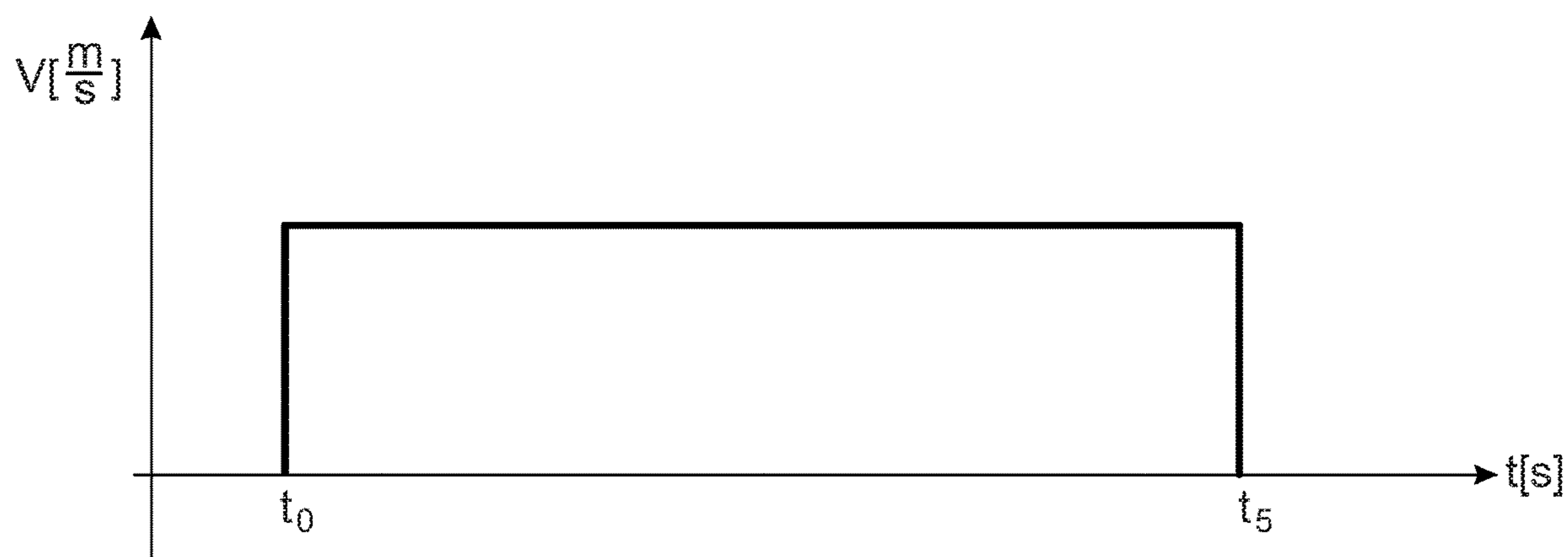


Fig. 17

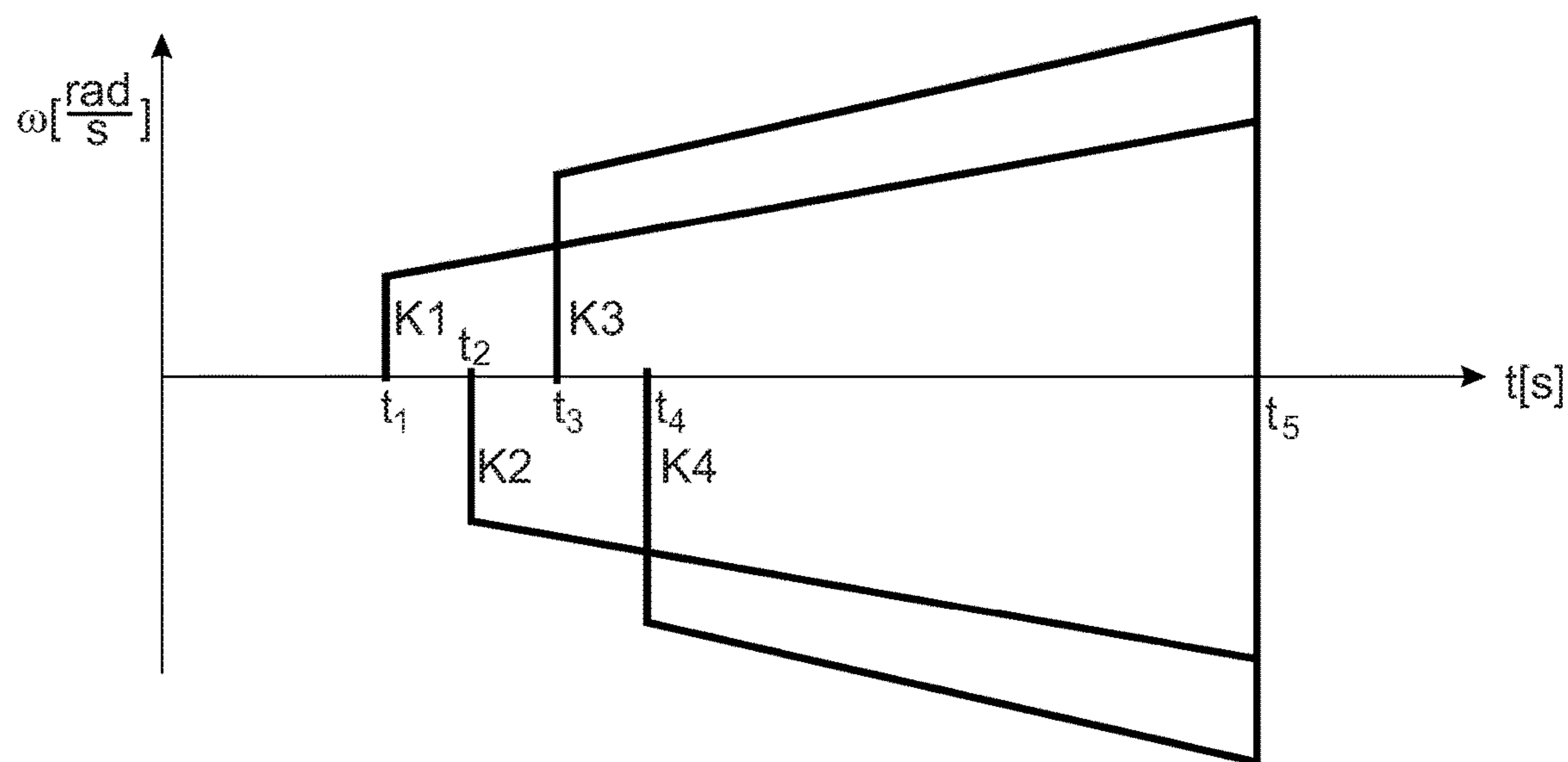
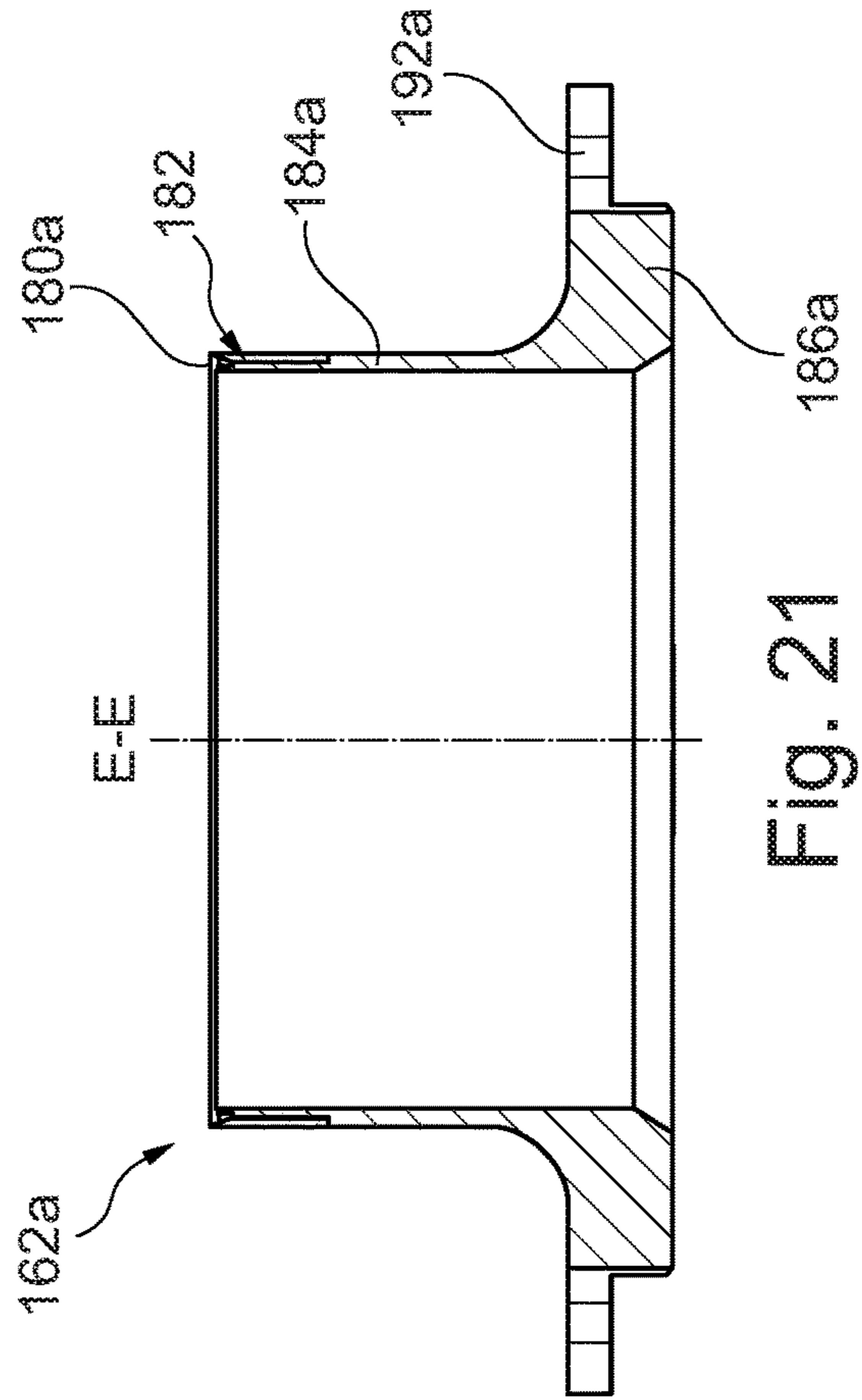
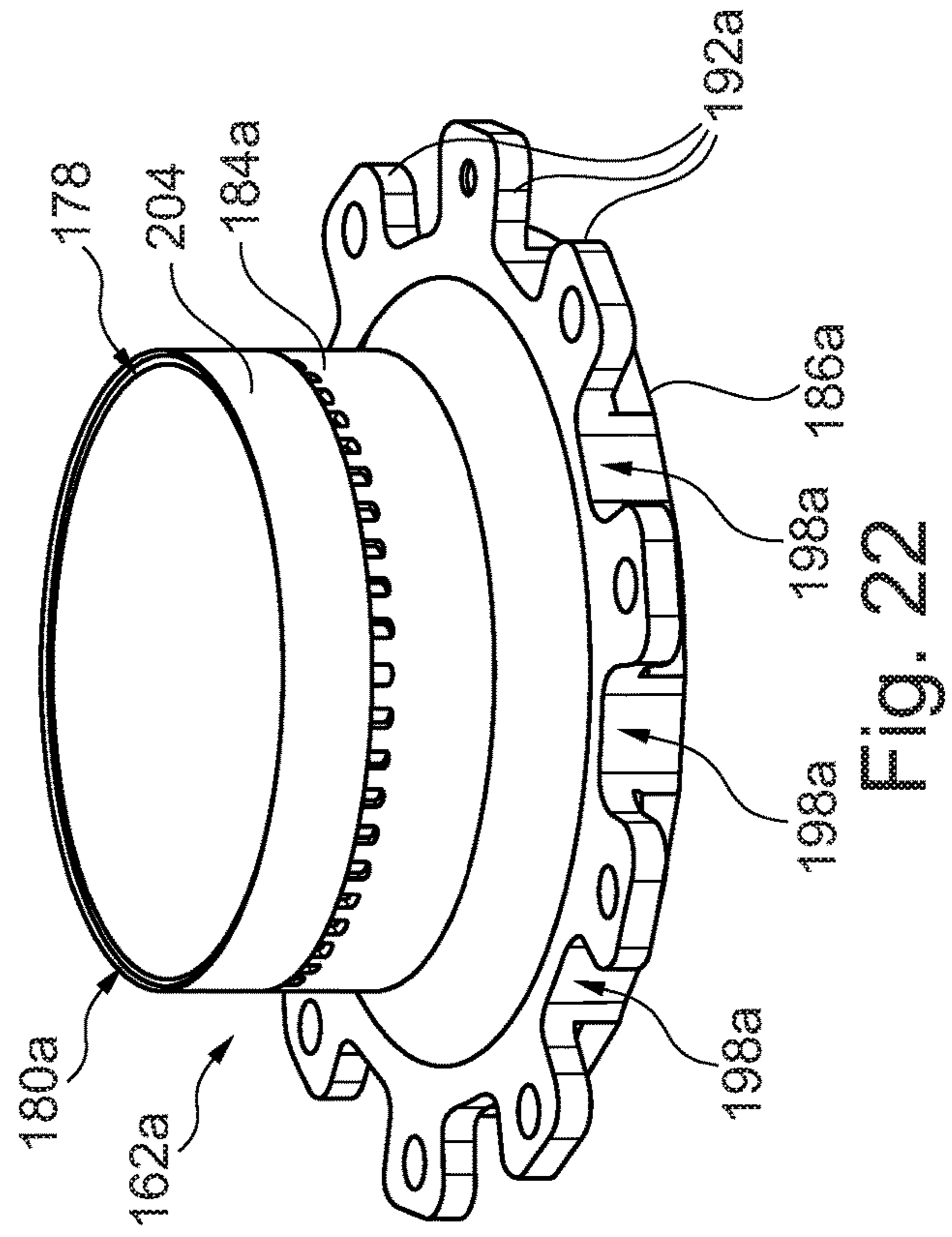
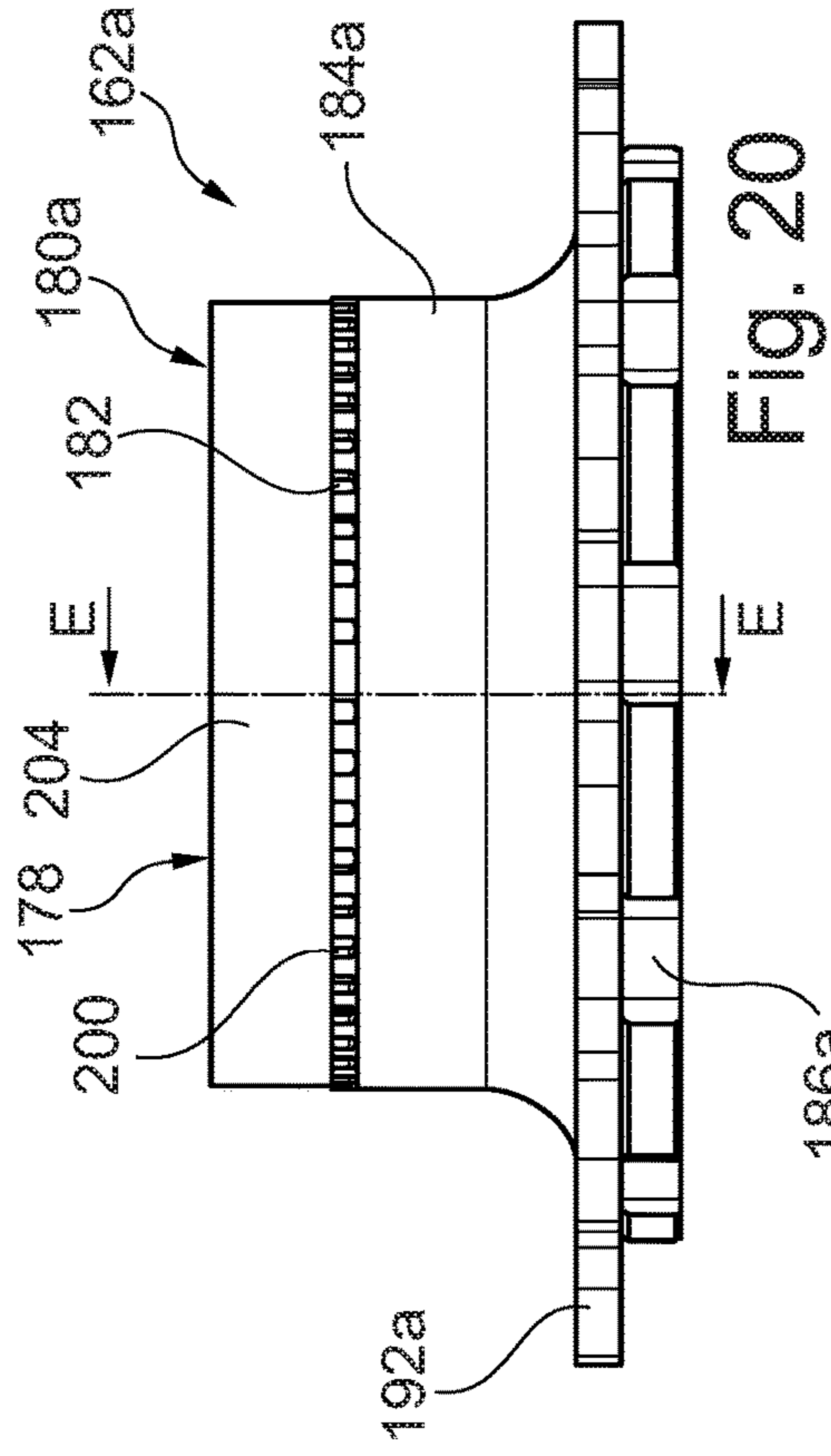
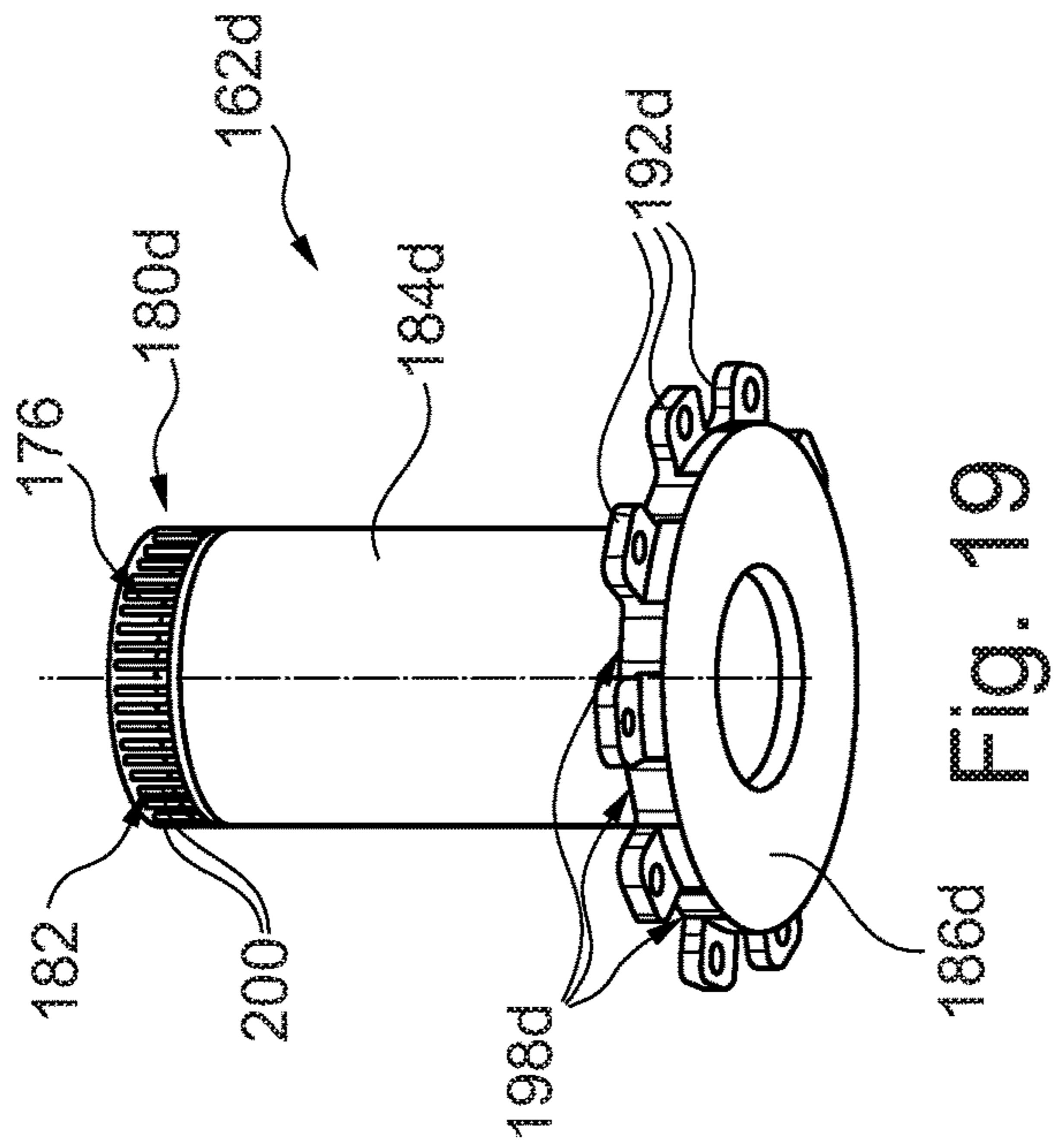


Fig. 18





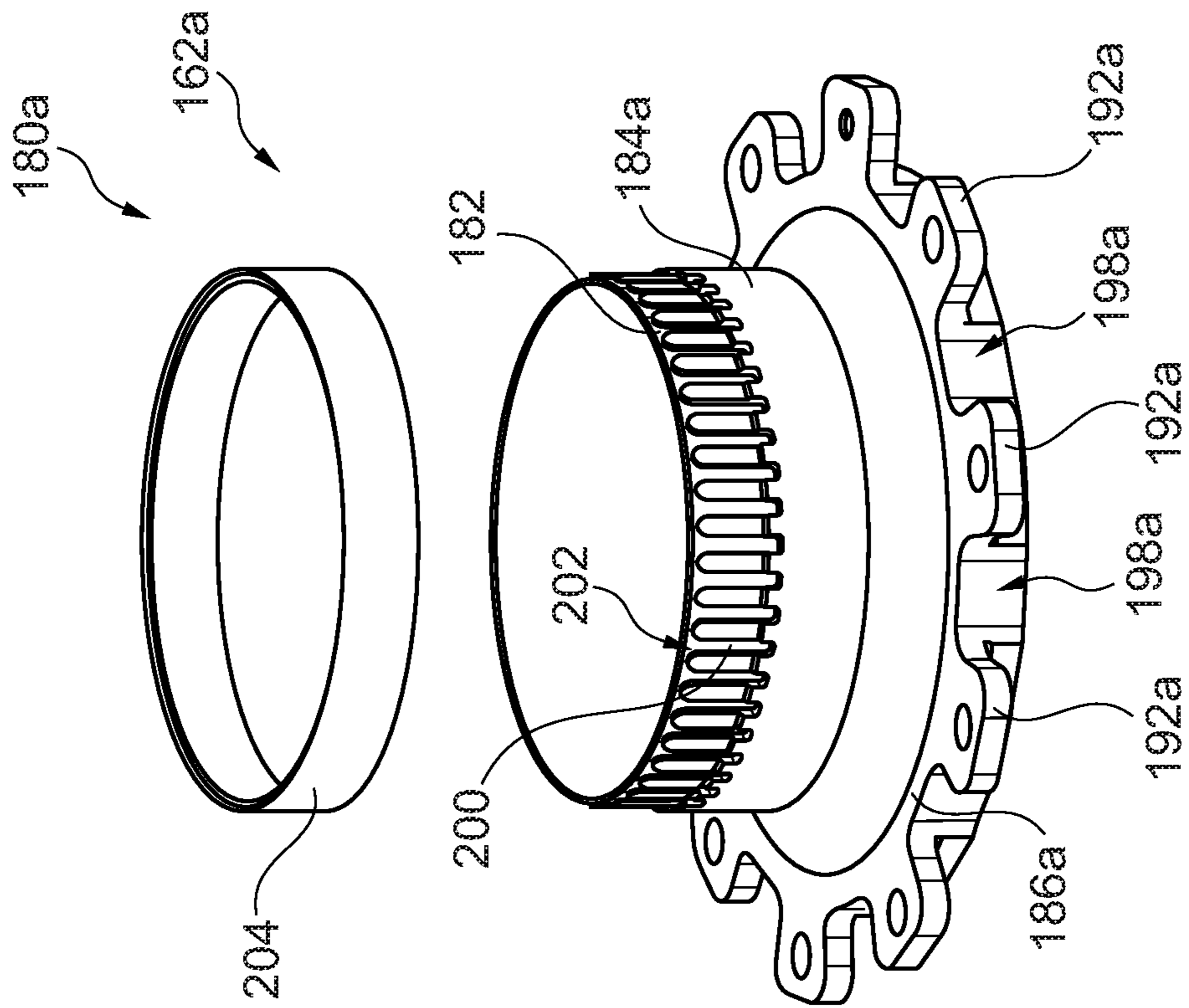


Fig. 24

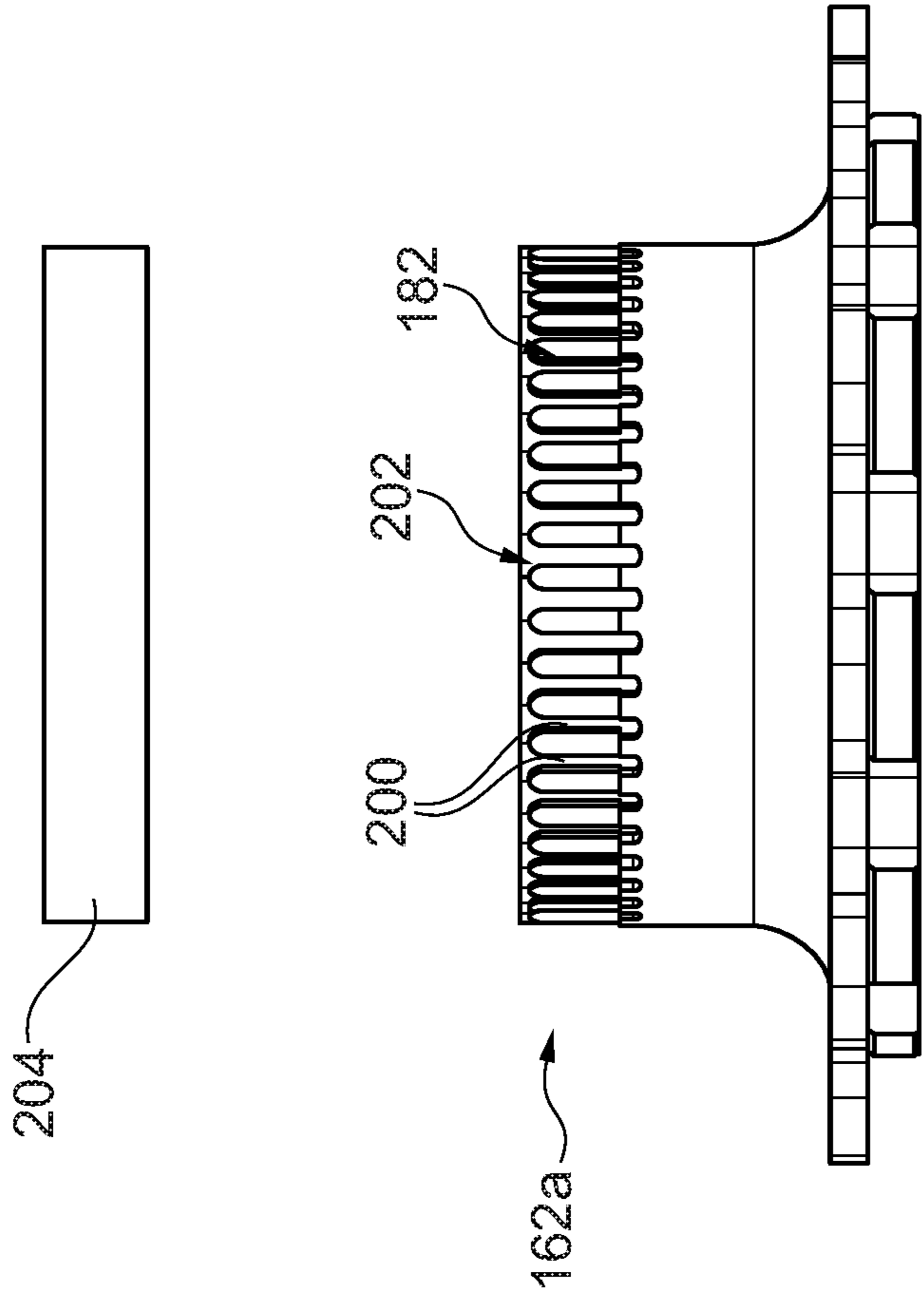


Fig. 23



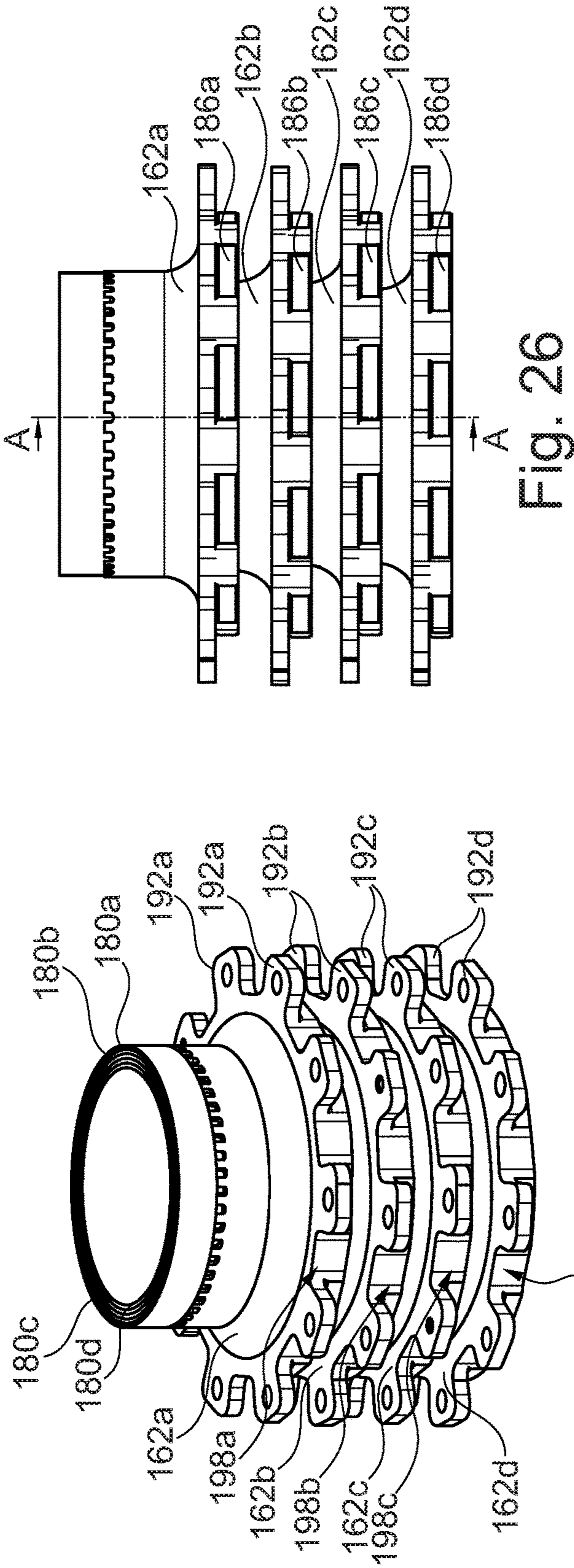


Fig. 26

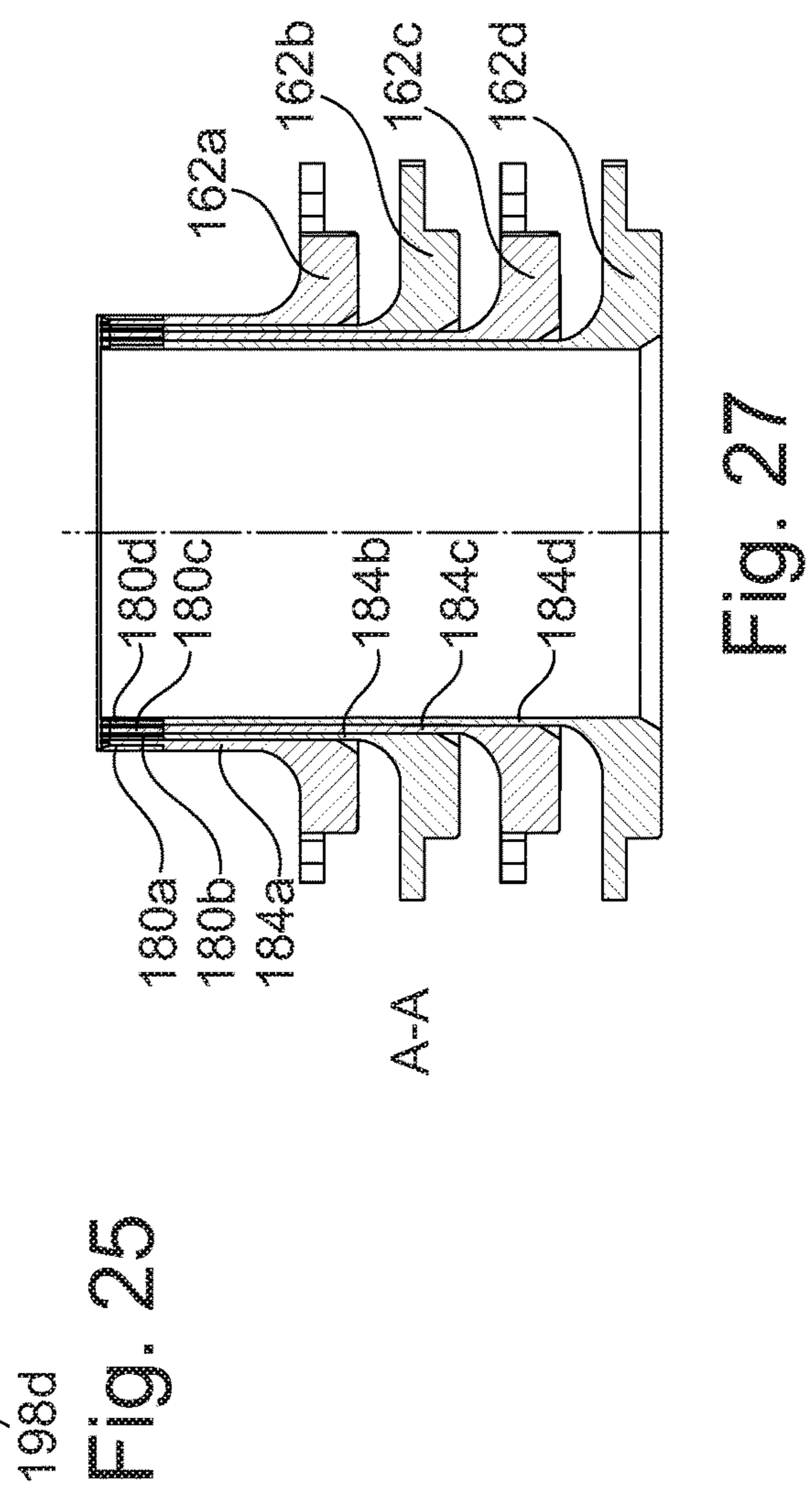
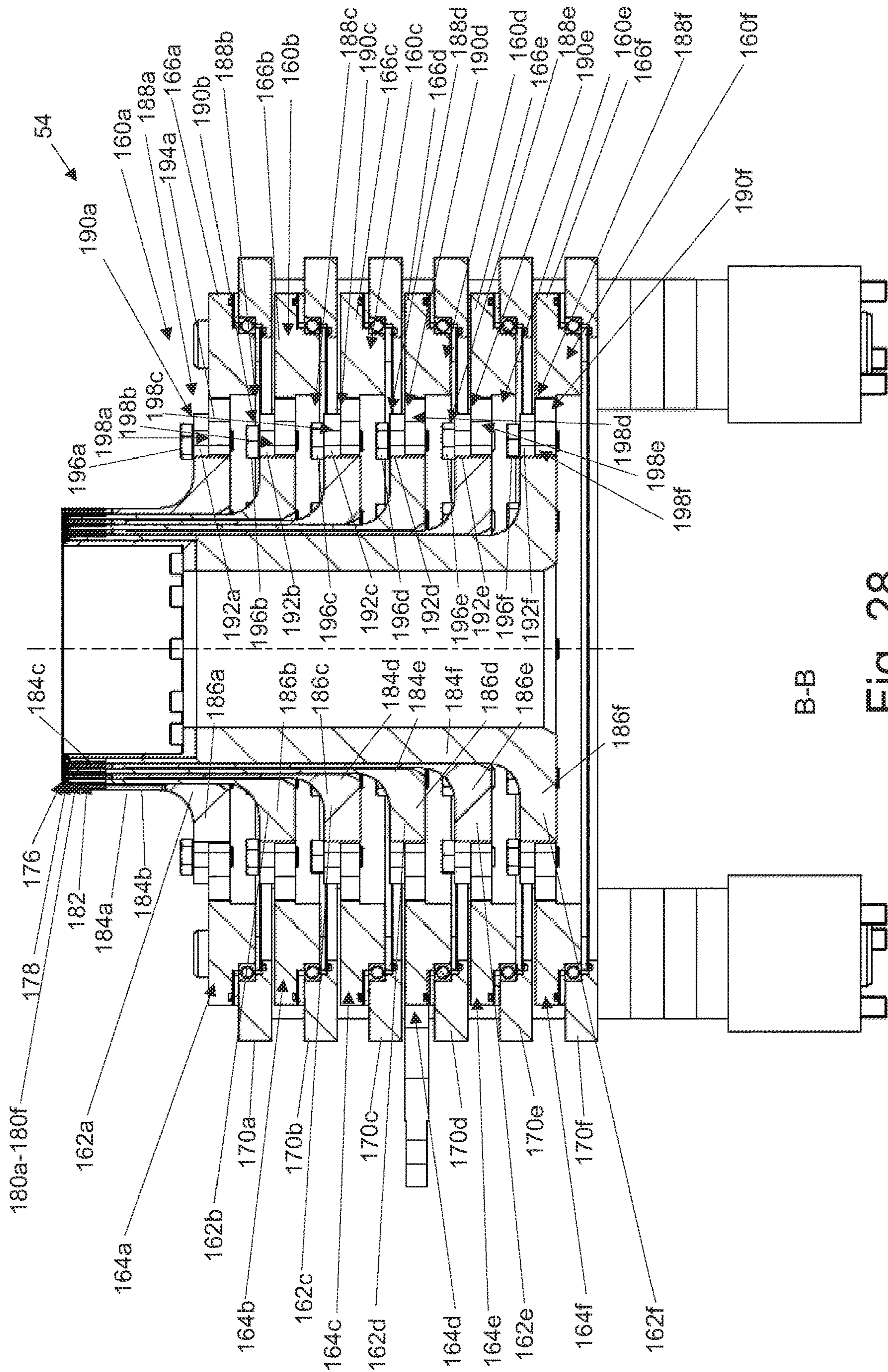


Fig. 27



B-B

Fig. 28



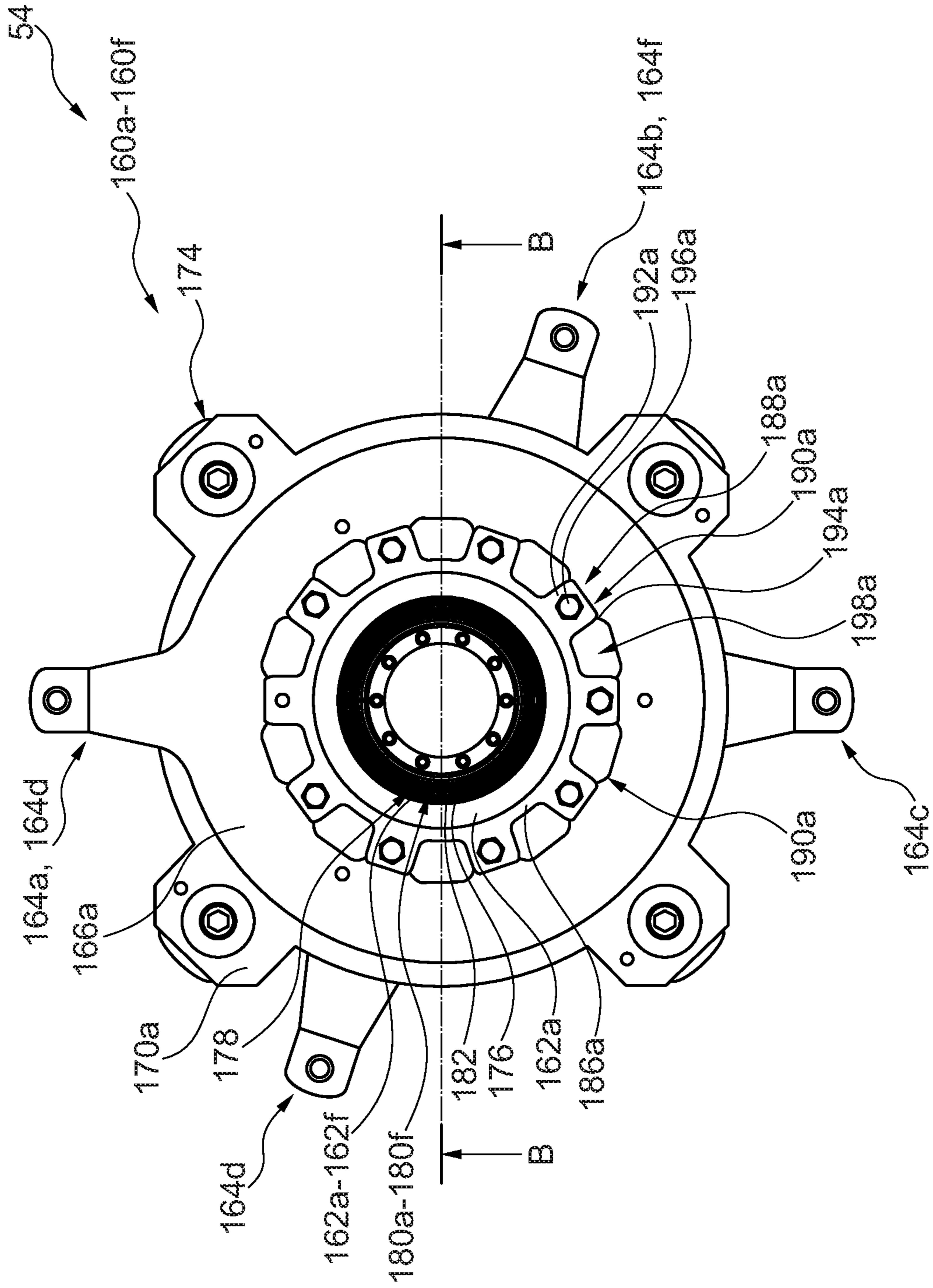
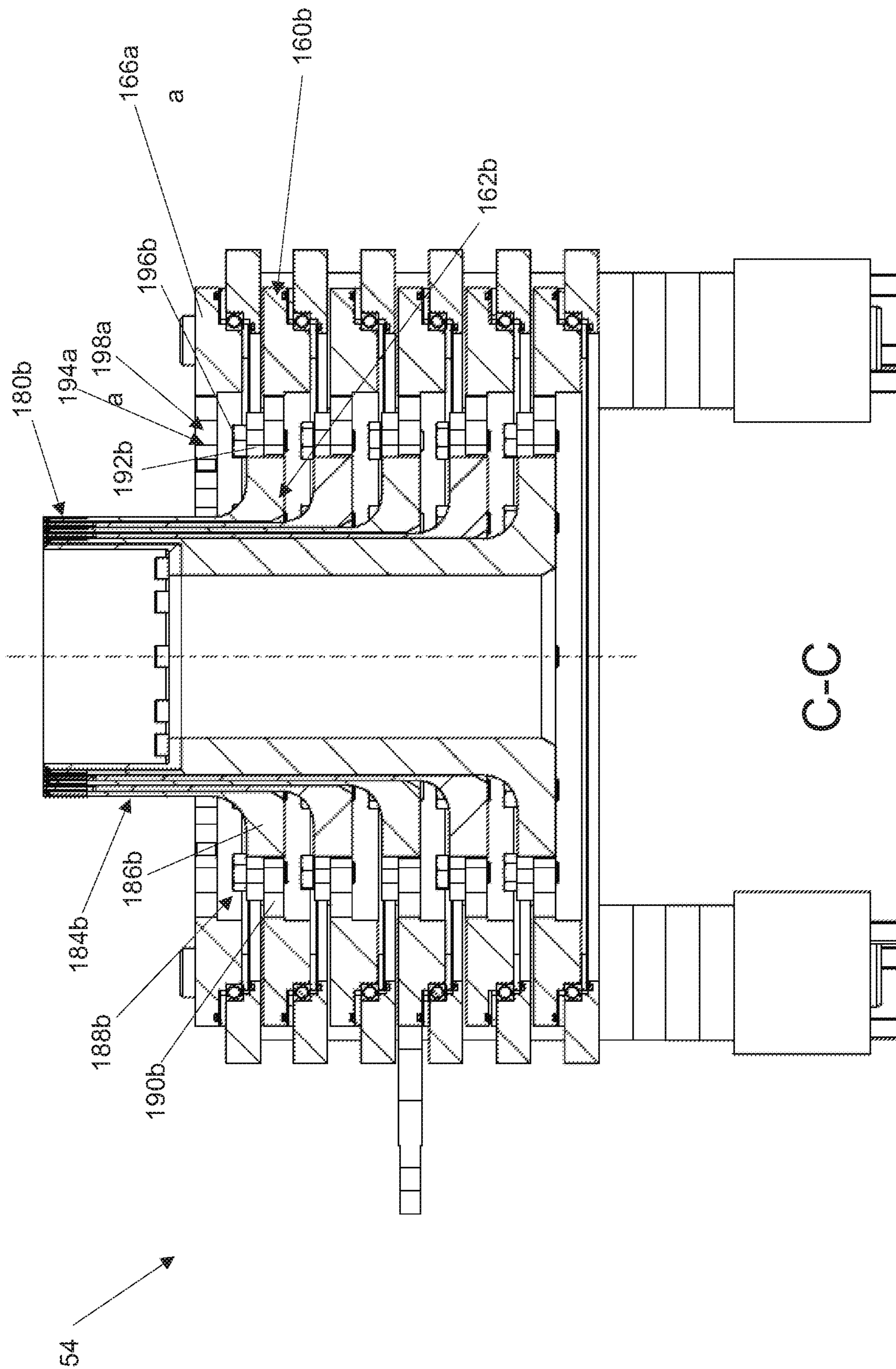


Fig. 29





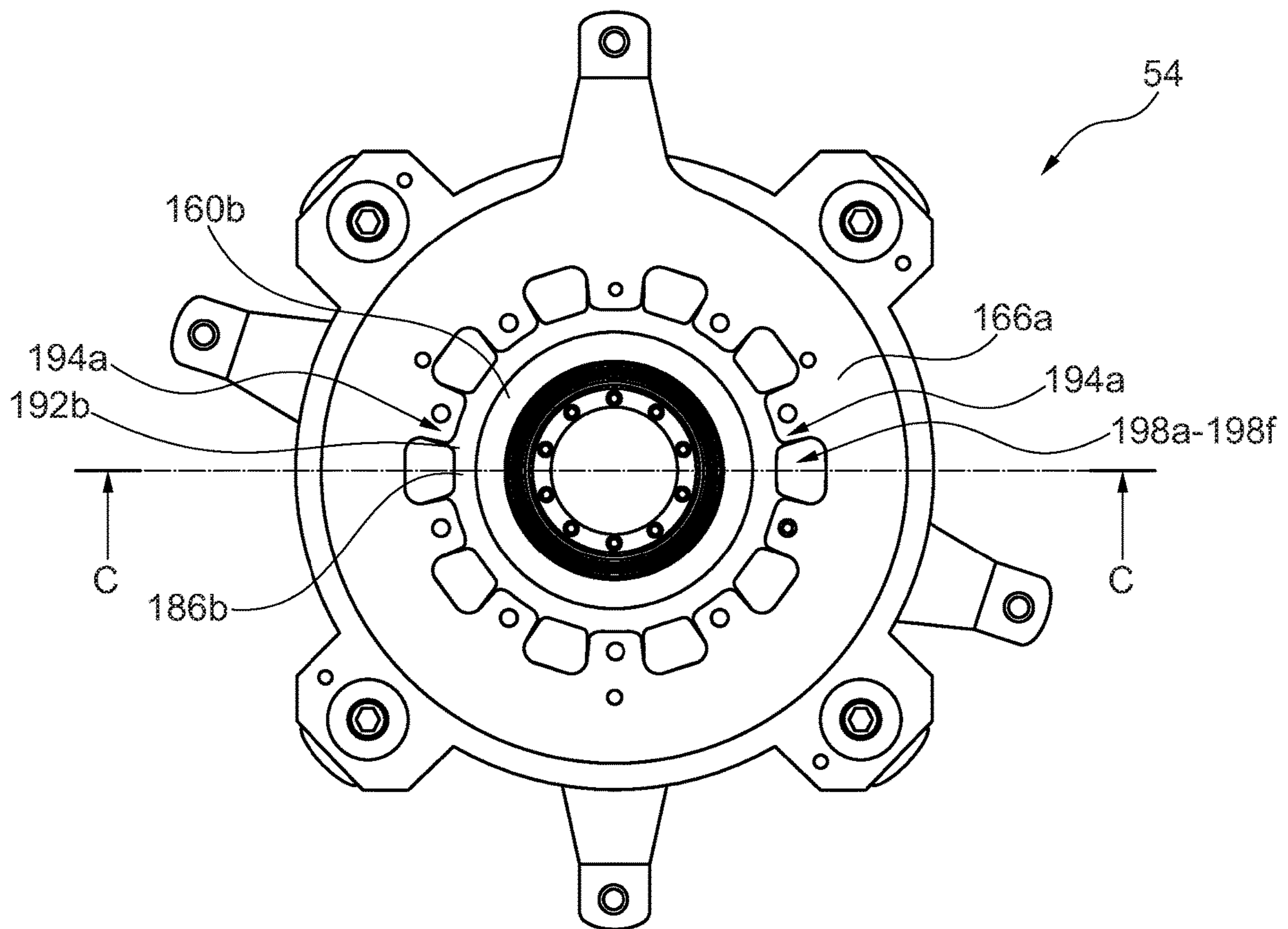
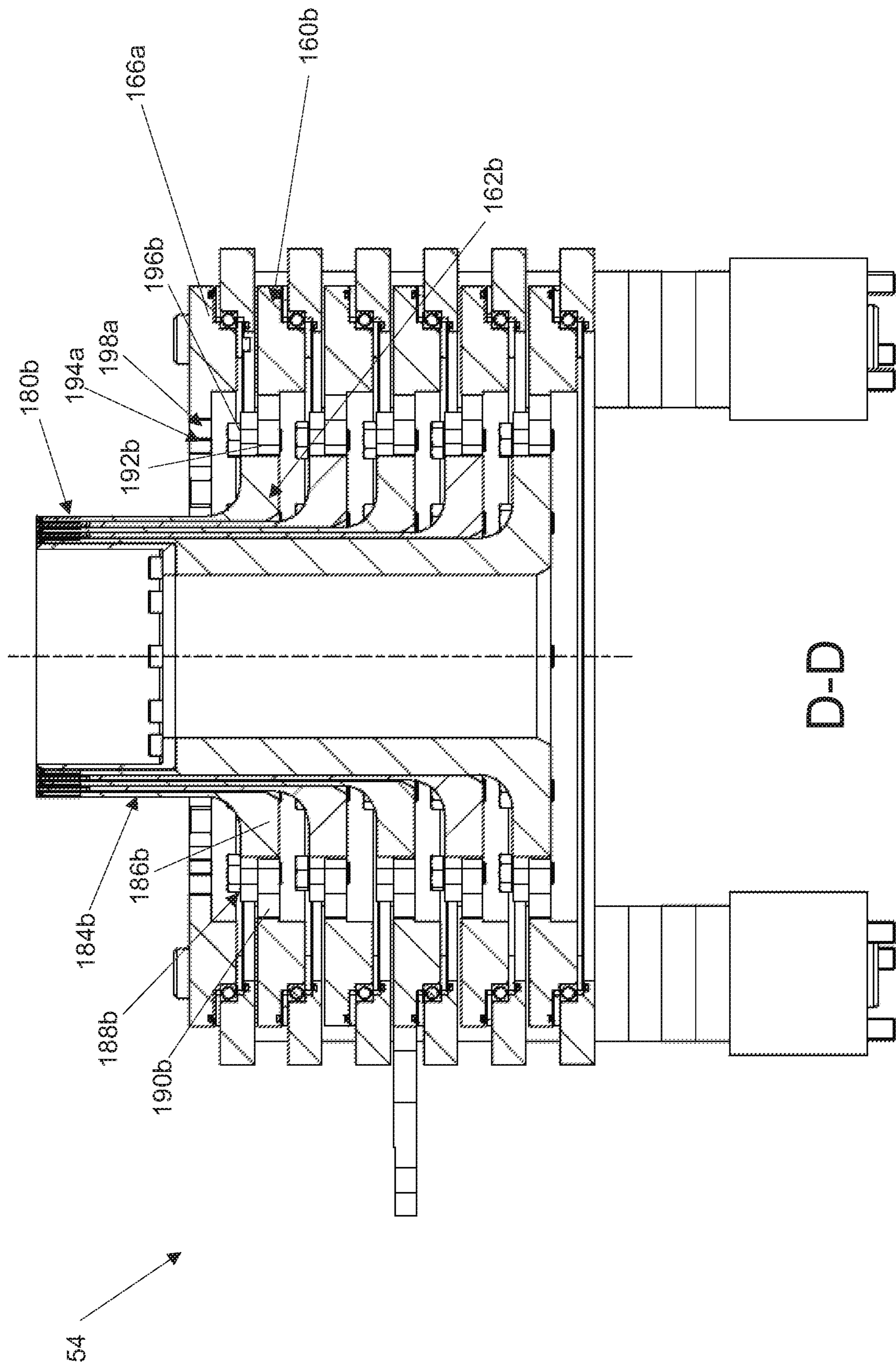


Fig. 31





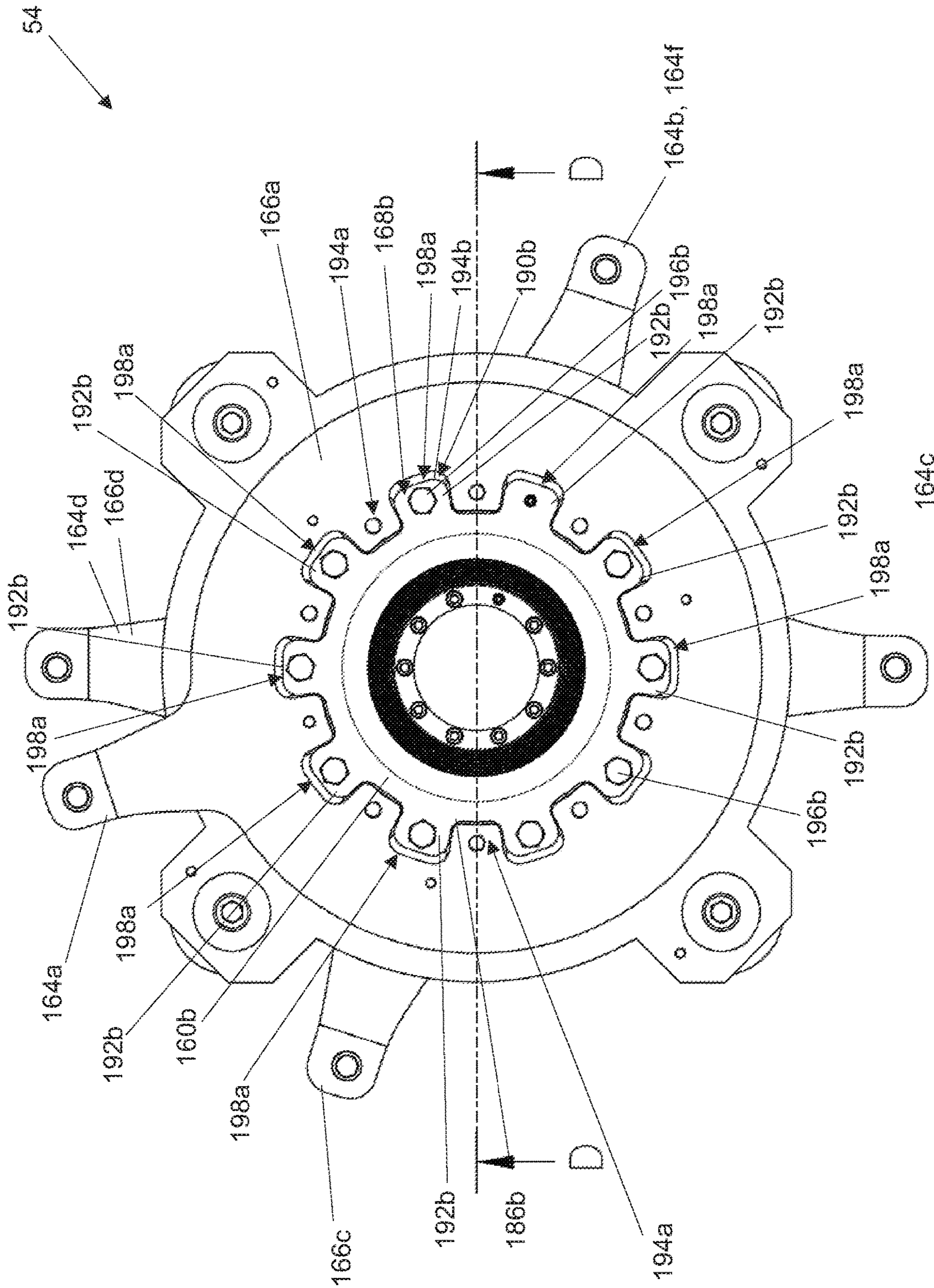


Fig. 33

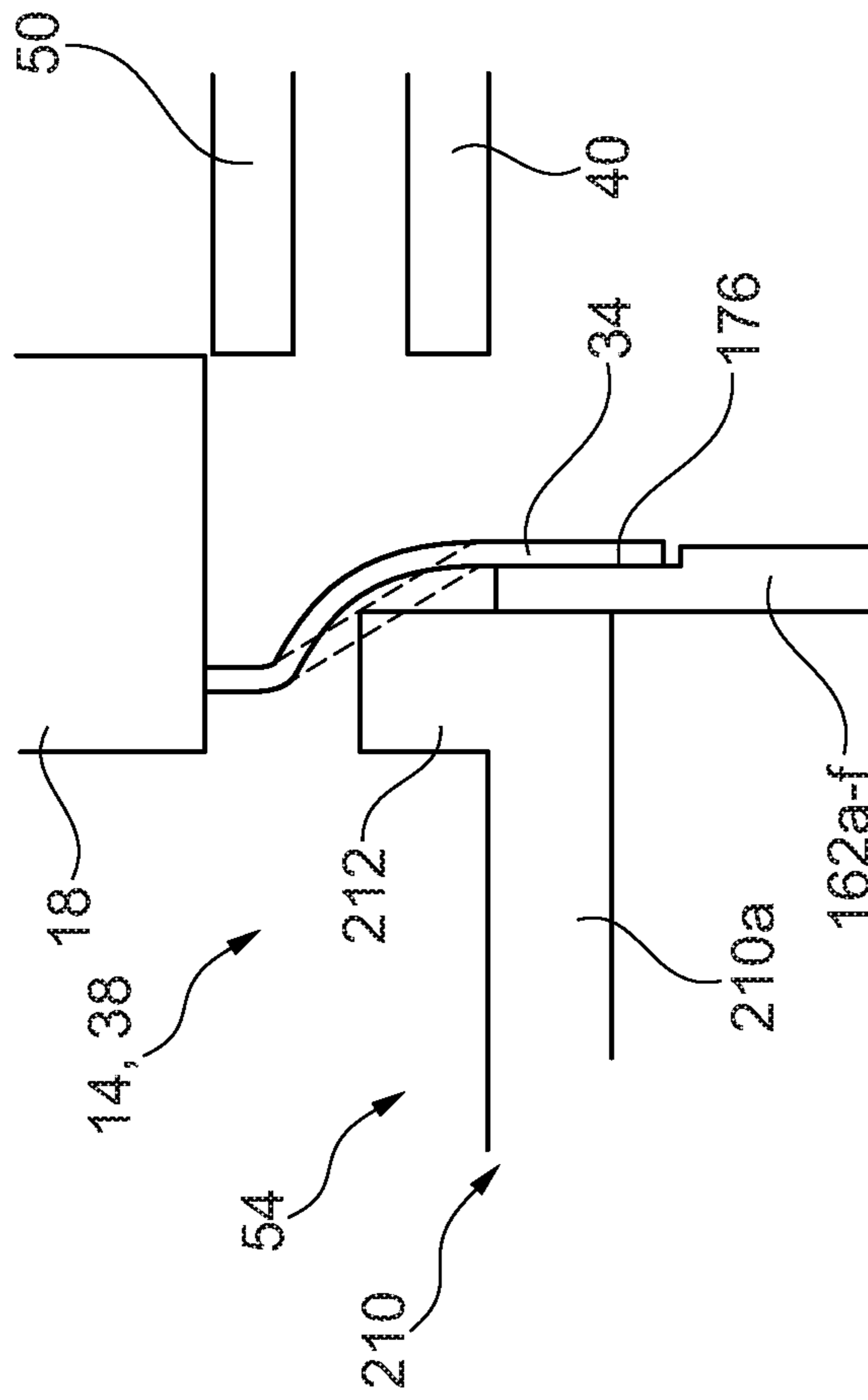


Fig. 34a

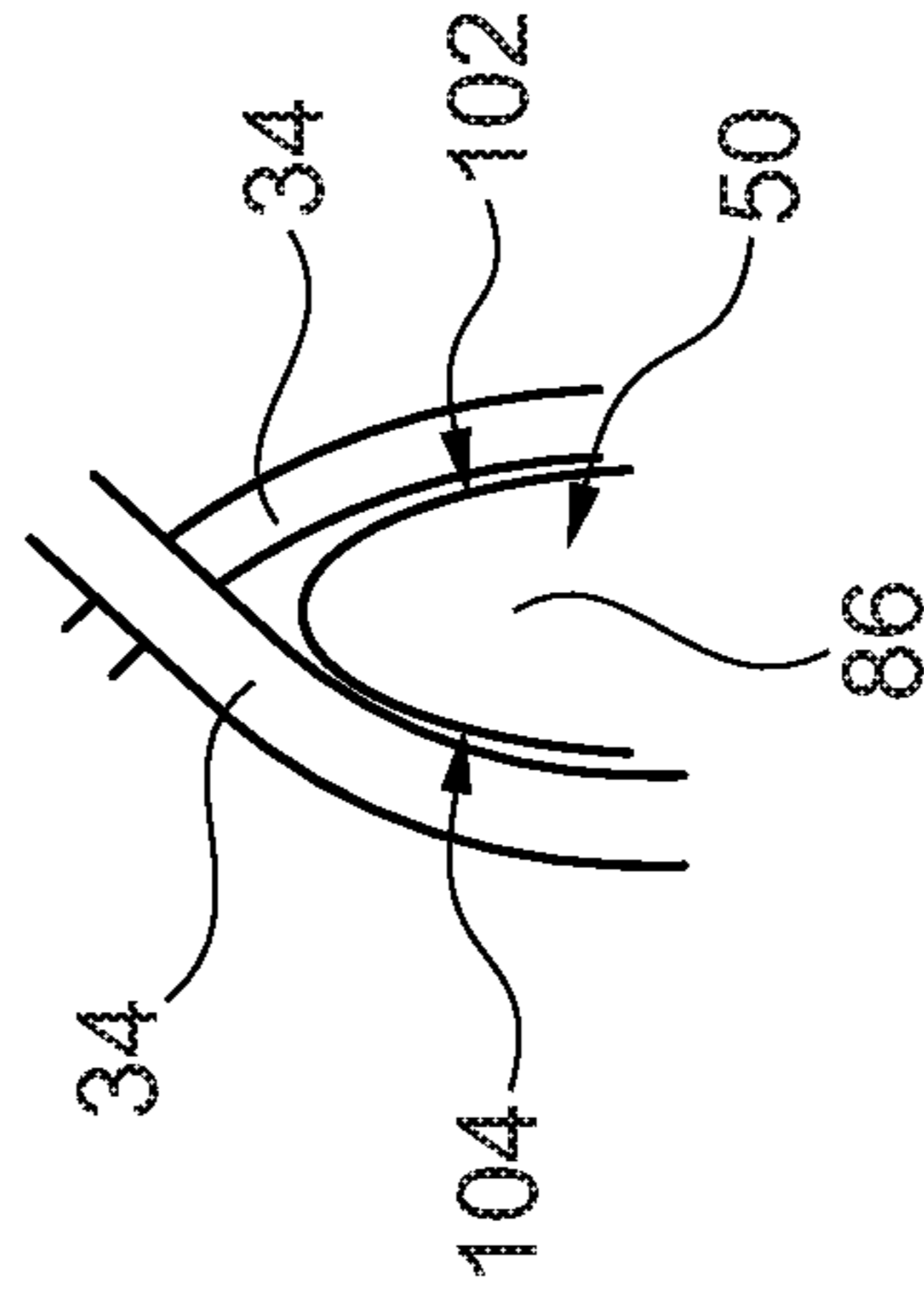


Fig. 34b

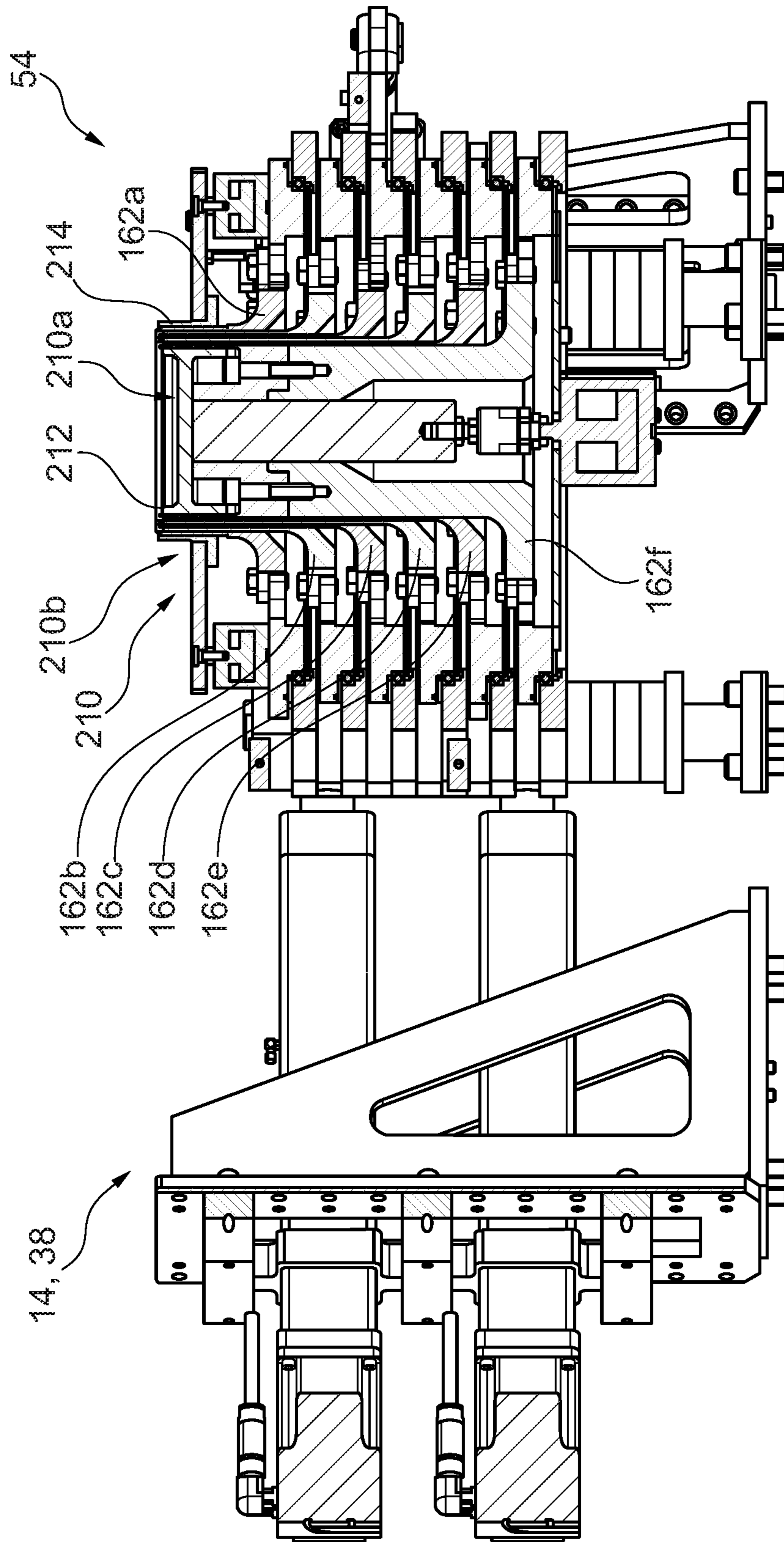


Fig. 35



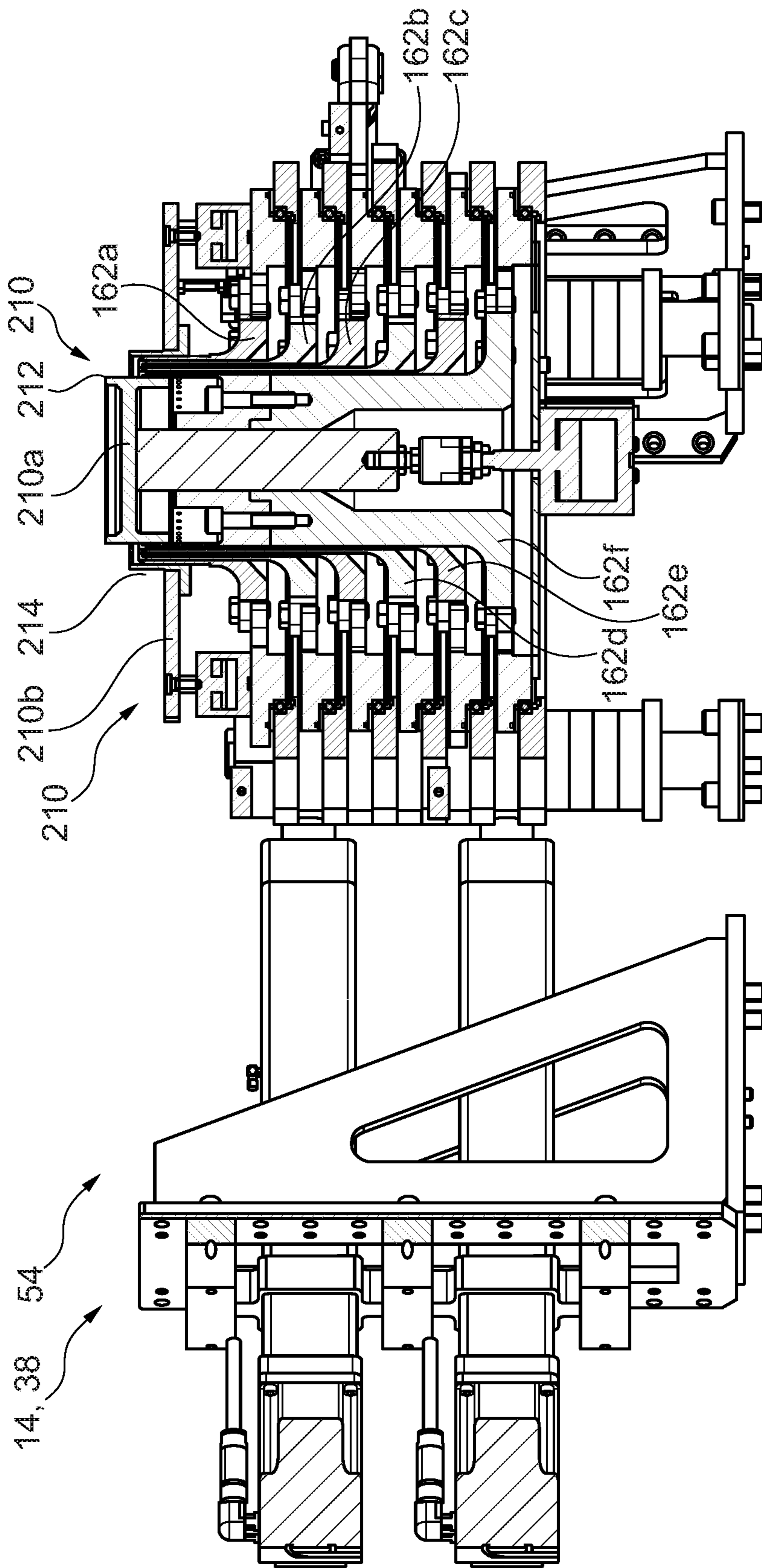


Fig. 36

**DEVICE AND METHOD FOR SHAPING  
WIRE ENDS IN A CIRCUMFERENTIAL  
DIRECTION**

CROSS-REFERENCES TO RELATED  
APPLICATIONS

This application claims the benefit of the International Application No. PCT/DE2019/100162, filed on Feb. 20, 2019, and of the German patent application No. 102018103929.8 filed on Feb. 21, 2019, the entire disclosures of which are incorporated herein by way of reference.

FIELD OF THE INVENTION

The invention relates to a wire end shaping device for use during production of a component, which is equipped with coils, of an electric machine, for shaping wire ends which protrude from an annular housing of the component. The invention furthermore relates to a component production device for producing a component, which is equipped with coils, of an electric machine, comprising a wire end shaping device of said type. Finally, the invention relates to a wire end shaping method, which can be performed preferably using such a wire end shaping device or component production device, for shaping wire ends, which protrude from an annular housing, during the course of production of a component, which is equipped with coils, of an electric machine.

BACKGROUND OF THE INVENTION

Electric machines are to be understood, in particular, to mean machines for converting electrical energy into kinetic energy and machines for converting kinetic energy into electrical energy. In particular, the term is to be understood as encompassing electric motors and generators.

In the production of machine elements of such electric machines, for example stators or rotors, it is often necessary for ends of conductors which are formed from wires to be connected to one another or jointly processed in some other way, for example jointly cut or shaped.

For example, there are electric motors in which coil windings, in particular of the stator, are formed from different wire pieces, the ends of which are then connected to one another. Devices and methods for connecting wire ends of hairpins in order to form stator windings of electric machines have already been proposed in which the wire ends are welded to one another. Here, devices and methods for positioning and bracing the wire ends before the welding process are provided.

In order to be able to correctly weld the wire ends, it is advantageous for the wire ends which protrude out of the individual grooves of the housing of the component, for example after the introduction of hairpins or the like, to firstly be shaped, in particular flared and set, for example in order to thus form wire end pairs which are to be connected to one another.

A wire end shaping device for deforming wire ends, which protrude out of a housing of a component of an electric machine, in a circumferential direction is known for example from U.S. Pat. No. 6,519,993 B2.

The problem addressed by the invention is that of providing devices and methods with which the wire shaping can be performed more reliably than has hitherto been known.

SUMMARY OF THE INVENTION

The invention provides, in accordance with a first aspect thereof, a wire end shaping device for use during production

of a component, which is equipped with coils, of an electric machine, for shaping wire ends which protrude from an annular housing of the component, comprising a bending device for bending wire ends in a circumferential direction, a relative movement device for moving the housing and the bending device in a relative manner in an axial direction, and a control unit,

wherein the bending device

has a first receiving and rotating means with a first receiving and rotating unit for receiving first wire ends and for turning the first wire ends for bending in a circumferential direction and with a first drive element for driving the first receiving and rotating unit, and

a second receiving and rotating means with a second receiving and rotating unit for receiving second wire ends, which are radially offset with respect to the first wire ends, and for turning the second wire ends for bending in the circumferential direction and with a second drive element for driving the second receiving and rotating unit,

wherein the first and second receiving and rotating unit are rotatable relative to one another and relative to the housing but are static relative to one another in an axial direction,

wherein the control unit is, for the bending of the wire ends, designed to activate the relative movement device to implement a relative movement between housing and the bending device and, during this, drive the first drive element with a first movement profile and the second drive element with a second movement profile, wherein the first and the second movement profile are configured to differ in a manner dependent on the relative movement such that an axial relative movement of the respective rotating receiving and rotating unit and end regions, engaging therein, of the wire ends is prevented.

In refinements of the invention, the different movement profiles are purely rotational movement profiles. The movement profiles comprise, in particular, purely rotational movements, wherein, as movement parameters, one or more of the parameters rotational directions, rotational speeds, rotational accelerations (in the positive or negative range), starting points, durations of rest or movement and end points may be set so as to vary over time or over the axial relative distance in a manner dependent on the respective bending situation.

It is preferable that the first receiving and rotating unit is designed for receiving and turning relatively long protruding first wire ends and the second receiving and rotating unit is designed for receiving and turning relatively short protruding second wire ends, and that the first movement profile comprises an earlier start of the rotational movement than the second movement profile.

It is preferable that the bending device has a third receiving and rotating means, with a third receiving and rotating unit for receiving third wire ends, which are radially offset with respect to the first and second wire ends, and for turning the third wire ends for bending in the circumferential direction, and a third drive element for driving the third receiving and rotating unit, wherein the control unit is designed to drive the third drive element with a third movement profile, which differs from the first and second movement profile, in a manner dependent on the relative movement between the bending device and the housing.

It is preferable that the bending device has a fourth receiving and rotating means, with a fourth receiving and rotating unit for receiving fourth wire ends, which are radially offset with respect to the first to third wire ends, and for turning the fourth wire ends for bending in the circum-



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ferential direction, and a fourth drive element for driving the fourth receiving and rotating unit, wherein the control unit is designed to drive the fourth drive element with a fourth movement profile, which differs from the first to third movement profile, in a manner dependent on the relative movement between the bending device and the housing.

It is self-evidently also possible for an even greater number of receiving and rotating means to be provided, with, in each case, one receiving and rotating unit which is rotatable in accordance with a dedicated movement profile. Preferably, the number of receiving and rotating means corresponds to the number of wire ends which protrude from each groove of the housing.

It is preferable that the control unit is designed to activate the relative movement device to implement the axial relative movement with a uniform speed, and to activate the drive elements such that at least one, multiple or all of the movement profiles cause a rotational movement with a varying rotational speed.

It is preferable that the control unit is designed to activate the relative movement device to implement the axial relative movement with a uniform speed, and to activate the drive elements such that at least one, multiple or all of the movement profiles cause a rotational movement with a gradually or uniformly changing rotational speed.

It is preferable that the control unit is designed to activate the relative movement device to implement the axial relative movement with a uniform speed, and to activate the drive elements such that at least one, multiple or all of the movement profiles cause a rotational movement with a rotational speed which increases or decreases in a continuous manner and/or to an increasing or decreasing extent.

It is preferable that the control unit is designed to activate the relative movement device to implement the axial relative movement with a uniform speed, and to activate the drive elements such that at least one, multiple or all of the movement profiles cause a rotational movement with a changing rotational speed such that an axial relative movement of the respective rotating receiving and rotating unit and end regions, engaging therein, of the wire ends is prevented.

It is preferable that the control unit is designed to activate the relative movement device to implement the axial relative movement with a uniform speed, and to activate the drive elements such that at least one, multiple or all of the movement profiles cause a rotational movement with movement parameters adapted to different lengths of the wire ends to be received in each case.

It is preferable that the receiving and rotating units are each formed by or have a setting crown.

It is preferable that bending edges formed on the receiving and rotating units are all situated substantially in the same plane extending in a radial and circumferential direction.

It is preferable that the wire end shaping device comprises a clamping device which is arrangeable between the housing and the bending device and which serves for fixedly holding the wire ends on the housing during the bending by the bending device.

It is preferable that the clamping device has a first clamping means, which is fixable to the housing or in a positionally static manner, and a second clamping means, which is movable relative to the first clamping means in order to clamp wire ends between the first clamping means and the second clamping means.

It is preferable that the clamping device is arranged on a housing holder which is rotatable and/or axially movable relative to the bending device.

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It is preferable that the clamping device has rounded and/or beveled bending formations or rounded and/or beveled bending edges for the abutment and shaping of the wire ends during the bending.

It is preferable that the clamping device has a first clamping means, which is fixable to the housing or in a positionally static manner, and a second clamping means, which is movable relative to the first clamping means in order to clamp wire ends between the first clamping means and the second clamping means.

It is preferable that the first clamping means is of disk-shaped and/or annular form.

It is preferable that the first clamping means is designed for support against inner wall regions of the housing, which inner wall regions are arranged between grooves of the housing. The wire segments which have wire ends can be arranged in said grooves.

It is preferable that the first clamping means has multiple segments which are radially movable for the purposes of fixing to inner wall regions of the housing, which inner wall regions are arranged between grooves of the housing, in which grooves the wire segments which have wire ends can be arranged.

It is preferable that the first clamping means has radially movable segments and an axially movable drive element for jointly driving the radial movement of the segments by means of a conical control surface.

It is preferable that the first clamping means has a first clamping region for abutment against radially inner sides of wire ends, arranged radially at the inside, of the wire ends protruding out of the grooves of the housing.

It is preferable that the clamping device has a number of radially movable clamping fingers for fixedly clamping the wire ends.

It is preferable that the clamping fingers are formed on the second clamping means.

It is preferable that the clamping fingers are formed on the second clamping means and are designed to clamp the wire ends to the first clamping means.

It is preferable that the clamping fingers are formed on the second clamping means and have, in each case, one engagement end for engaging into a corresponding complementary cutout on the first clamping means. Preferably, the engagement end is received in the cutout with such a degree of play that easy insertion and removal of the clamping finger in a radial direction is made possible. Preferably, the respective engagement ends, when in the state in which they have been inserted into the cutout, can be supported on the cutout in order to support the clamping fingers against a displacement in an axial direction and/or circumferential direction.

It is preferable that the clamping device have rounded bending formations for the shaping at the wire ends during the bending process. Preferably, the clamping device has a group of first rounded bending formations for the shaping during bending of wire ends in a radial direction and has a group of second rounded bending formations for the shaping during bending of the wire ends in a circumferential direction.

The bending formations are preferably formed as rounded bending edges.

It is preferable that at least one of the bending formations is formed on each of the clamping fingers. Preferably, at least one first bending formation for the shaping during bending of the wire ends in the radial direction and at least one second bending formation for the shaping during the bending in a circumferential direction are provided on each clamping finger. Preferably, the first and the second bending



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formations are designed as edges which adjoin one another at a corner, in particular, edges of a step or receiving groove on the clamping finger for the wire ends.

It is preferable that the clamping fingers are formed in a number which corresponds to the number of groups of wire ends which are to be clamped together.

It is preferable that the clamping fingers are formed in a number which corresponds to the number of grooves of the housing of the component to be produced.

It is preferable that the clamping fingers are each designed for jointly clamping all of the wire ends which jointly protrude from the associated groove of the housing.

It is preferable that the clamping fingers are guided on a holding ring so as to be radially displaceable in a jointly or synchronously driven manner.

It is preferable that the clamping fingers each have a second clamping region, formed at a step, for abutment against a radially outer side of the radially outer wire end of the wire ends which protrude from the associated groove of the housing.

It is preferable that the clamping fingers each have a conically tapering tip at the free end.

It is preferable that the clamping fingers have a step at their free ends such that a receiving groove for two, three, four, five, six or more wire ends which jointly protrude from the associated groove of the housing is formed between adjacent clamping fingers situated in a clamping position.

It is preferable that the receiving groove is delimited, at one side running substantially in a radial direction and at one side running substantially in a circumferential direction, by one clamping finger, and at another side running substantially in a radial direction, by the other clamping finger, and is open at another side running substantially in a circumferential direction.

It is preferable that the bending device has at least one radial support means for radially supporting wire ends during the shaping.

It is preferable that the bending device has an inner radial support means for radially supporting wire ends on a radially inner side.

It is preferable that the bending device has an outer radial support means for radially supporting wire ends on a radially outer side.

It is preferable that the at least one radial support means is designed as a sleeve.

It is preferable that the at least one radial support means or at least one or more of multiple radial support means is axially movable.

It is preferable that the at least one radial support means has a flexible support region.

It is preferable that the at least one radial support means is movable such that the movement stops when an abutment is reached.

It is preferable that the controller is designed so as to activate the at least one radial support means such that the at least one radial support means, before the shaping of the wire ends, moves axially out from a retracted position into a supporting position and, after the shaping of the wire ends, moves axially back in from the supporting position into the retracted position.

According to a further aspect, the invention provides a component production device for producing a component, which is equipped with coils, of an electric machine, comprising a wire end shaping device according to any of the preceding refinements.

According to a further aspect, the invention provides a wire end shaping method, which can be and/or is to be

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performed during the course of the production of a component, which is equipped with coils, of an electric machine, for shaping wire ends which protrude from an annular housing of the component, comprising the following steps:

a) providing or using a bending device which has a first receiving and rotating unit, rotatable about an axis of rotation, for receiving first wire ends and which has a second receiving and rotating unit, rotatable about an axis of rotation, for receiving second wire ends which are radially offset with respect to the first wire ends, wherein the receiving and rotating units are rotatable relative to one another but are static or held relative to one another in an axial direction, and

b) moving the housing and the bending device in a relative manner in order to introduce the first wire ends into the first receiving and rotating unit and the second wire ends into the second receiving and rotating unit and in order to compensate an axial change in length of wire ends during a bending operation,

c) rotating the first receiving and rotating unit relative to the housing in order to bend the first wire ends after end regions of the first wire ends have been introduced into the first receiving and rotating unit, and rotating the second receiving and rotating unit relative to the housing in order to bend the second wire ends after end regions of the second wire ends have been introduced into the second receiving and rotating unit, wherein the rotation of the first and second receiving and rotating units is performed with different movement profiles in a manner dependent on the axial relative movement and on the axial length of the wire ends such that an axial relative movement between the end regions of the wire ends and the respective receiving and rotating unit is prevented.

It is preferable that the axial relative movement in step b) is performed with a uniform speed and the rotation in step c) is performed in each case with a varying rotational speed.

It is preferable that, in step a), the bending device with a third receiving and rotating unit for receiving third wire ends, which are radially offset relative to the first and second wire ends, and for turning the third wire ends for the purposes of bending in the circumferential direction is provided or used, wherein the third receiving and rotating unit is rotatable relative to the first and second receiving and rotating unit about the axis of rotation but is axially static or held and, in step c), after the introduction of end regions of the third wire ends, is rotated such that a relative movement of the end regions in the third receiving and rotating unit is prevented.

It is preferable that, in step a), the bending device with a fourth receiving and rotating unit for receiving fourth wire ends, which are radially offset relative to the first to third wire ends, and for turning the fourth wire ends for the purposes of bending in the circumferential direction is provided or used, wherein the fourth receiving and rotating unit is rotatable relative to the first to third receiving and rotating unit about the axis of rotation but is axially static or held and, in step c), after the introduction of end regions of the fourth wire ends, is rotated such that a relative movement of the end regions in the fourth receiving and rotating unit is prevented.

The first and second wire ends and possibly the third and/or fourth wire ends may protrude with an equal length, and the movement profiles may each correspondingly, in a manner dependent on the equal length, be configured differently in accordance with the different bend angles to be implemented. In another method approach, it is preferable that the wire ends protrude with different lengths, in particular such that wire ends situated radially further to the



outside have, at the start, a greater axially protruding length than wire ends situated radially further to the inside. In such a case, the movement profiles are configured to differ in a manner dependent on the different bend angles to be implemented and on the different protruding lengths.

It is preferable that the rotational movements of the individual receiving and rotating units are correspondingly, in the case of different lengths of the wire ends to be bent in each case, started at correspondingly different points in time.

It is preferable that the rotational movements of all receiving and rotating units and the relative movement between housing and bending device are ended at the same time.

A preferred refinement of the wire end shaping method comprises the following step:

d) clamping the wire ends, by means of a clamping device arranged between the housing and the bending device, during the bending process.

Step d) preferably comprises the following step:

fixing a first clamping means to the housing or in a positionally static manner, and moving a second clamping means relative to the first clamping means in order to clamp the wire ends between the first and the second clamping means.

Step d) preferably comprises the following step:

jointly clamping all wire ends which protrude out of a groove of a housing.

Step d) preferably comprises the following step:

supporting a first of multiple clamping means, which are movable relative to one another, against inner wall regions of the housing, which inner wall regions are arranged between grooves of the housing.

Step d) preferably comprises the following step:

radially moving multiple segments of a first of multiple clamping means, which are movable relative to one another, in order to fix the first clamping means to the housing and/or for the purposes of release from the housing.

Step d) preferably comprises the following step:

driving, by means of an axially movable drive element, a radial movement of multiple segments of a first of multiple clamping means, which are movable relative to one another, in order to fix the first clamping means to the housing and/or for the purposes of release from the housing, and jointly transmitting the axial movement to the segments by means of a control surface or a conical surface.

Step d) preferably comprises the following step:

abutting a first clamping region, which is formed on a first of multiple clamping means, which are movable relative to one another, against radially inner sides of wire ends, arranged radially at the inside, of the wire ends protruding out of the grooves of the housing.

Step d) preferably comprises the following step:

radially moving a number of clamping fingers, provided correspondingly to the number of groups of wire ends to be clamped together and/or of grooves in the housing, in order to clamp the wire ends or for the purposes of releasing the clamping.

Step d) preferably comprises the following step:

clamping in each case one group of the wire ends between a first clamping means and in each case one radially movable clamping finger of a second clamping means.

Step d) preferably comprises the following step:

supporting clamping fingers, which are moving into a clamping position, against displacement in a circumferential direction or in an axial direction.

Step d) preferably comprises the following step:

supporting clamping fingers, which are moving into the clamping position, against displacement in a circumferential direction or in an axial direction by means of positively locking engagement of each clamping finger into one or on a complementary grasping unit.

Step d) preferably comprises the following step:

supporting clamping fingers, which are moving into the clamping position, against displacement in a circumferential direction or in an axial direction by means of positively locking engagement of a tip of each clamping finger of a second clamping means into a corresponding tip receptacle groove on a first clamping means.

Step d) preferably comprises the following step:

deforming the wire ends by abutment against bending formations or bending edges arranged on the clamping device.

Step d) preferably comprises the following step:

jointly and/or synchronously driving the radial movement of clamping fingers in order to clamp the wire ends.

Step d) preferably comprises the following step:

jointly guiding the radial movement of clamping fingers on a holding ring.

Step d) preferably comprises the following step:

receiving the two, three, four, five, six or more wire ends, which are to be clamped and which jointly emerge from a groove of the housing, in a receiving groove which is formed by adjacent clamping fingers at the free ends thereof, wherein the receiving groove is delimited, at one side running substantially in a radial direction and at one side running substantially in a circumferential direction, by one clamping finger, and at another side running substantially in a radial direction, by the other clamping finger, and is open at another side running substantially in a circumferential direction.

Step c) preferably comprises the following step:

c1) radially supporting at least some of the wire ends on a radially inner side and/or on a radially outer side during the bending process.

Step c1) preferably comprises the following steps:

axially moving at least one radial support means out from a retracted position into a supporting position before the bending process and axially moving the at least one radial support means in from the supporting position into a retracted position after the bending process.

A preferred field of use for the above-described devices, methods and uses according to the invention is the production of components, which are equipped with coils, of an electric machine, and, more particularly, the production of stators of electric machines, and, more particularly, the production of stators of electric motors which as traction motors for electric motor vehicles, in particular in the power range from 10 kW to 400 kW. The devices, the method and use of the present invention, and of the advantageous refinements thereof, are suitable and usable, in particular, in the field of the production of electric motors or other electric machines, for example generators, which are configured for high power, reliable operation and high efficiency. In particular, it is the intention to produce electric motors which are usable as traction motors of electric vehicles or hybrid vehicles and which have, for example, a nominal power of between 20 kW and 400 kW. For the construction of stators of such powerful electric machines, it is advantageous to provide as high a coil density as possible. In particular, for this purpose, so-called hairpins are inserted into grooves of a stator housing, which grooves have previously been lined with a groove insulator, in particular groove insulating paper. Subsequently, the free wire ends are shaped in a radial



direction—flared/“flaring”—and shaped in a circumferential direction—“setting”—in order to form pairs of wire ends which are to be connected, in particular welded, to one another, such that closed coil windings can then be produced. Preferably, for this purpose, use is made of wires which are of not circular but rather rectangular form in cross section. The devices and methods and uses proposed here have the advantage that the bending can be performed in a more exact and more controlled manner.

In particular, the invention relates to a device for the shaping, directed in a circumferential direction, of bar-type conductors in electric machines.

During preferred production of stators, the conductors are firstly produced. These conductors are subsequently fitted into a laminated core. In the case of hairpins being used to produce the coils, wire ends then protrude at one end of the stator housing formed as a laminated core. Here, two, three, four or even more wire ends may also protrude closely adjacently to one another in each groove of the housing. For the further processing, it is advantageous for these conductor ends to be flared, for which purpose they are preferably firstly shaped in a radial direction. It is the intention for this process to be performed in an automated manner.

For the subsequent (automated) processes (in particular, setting and connecting of the wire ends to form coil windings), it is advantageous for the conductor ends to be shaped such that the position lies within a predefined tolerance range. From experience, the conductor ends can be particularly effectively shaped in a reproducible manner if they are braced at their free end and at the “foot” (top edge of the laminated core). The radial shaping of the conductor ends results in a shortening of the axial length. To avoid damage to the conductor ends, it is advantageous for the length compensation to be counterbalanced by means of a translational movement of the stator/shaping tool.

Preferred devices and methods according to the present invention can be particularly advantageously used on housings with the wire ends that have correspondingly been flared as described above. Preferred refinements of the invention are then preferably used for shaping the wire ends in a circumferential direction, preferably such that pairs of wire ends which are to be connected to one another can be formed, in order to thus close coil windings.

In combination with the clamping device according to one of the refinements mentioned above, a preferred refinement of the wire end shaping device makes possible the reproducible shaping of conductor ends, in particular by means of the individual processing of each conductor end.

The invention relates, in particular, to methods and devices for bending end portions of bar-type conductors, in particular of bar-type windings of electric machines.

In order to make the electrical interconnection of stators possible, in one process step in one advantageous refinement, the end portions of bar-type conductors are twisted. For this process step, the individual pins are firstly produced and fitted into a laminated core. In the case of a small spacing between the end portions of the bar-type conductors after the joining process, said end portions are preferably flared in a subsequent step in order to make precise twisting (setting) possible. Preferably, the end portions of the bar-type conductors are, for the flaring, shaped in a radial direction in order to produce a spacing between the individual conductors (flaring process).

In the case of stators, multiple conductors are arranged in one groove. Preferably, all conductors which lie on one radius after the flaring process are processed using one tool (in particular, arranged on a receiving and rotating unit).

During this processing, all conductor ends are preferably firstly introduced into pockets of the tool and subsequently twisted (rotation of the tool about a central axis). This twisting causes a shortening of the bar-type conductors in an axial direction. To prevent a relative movement from occurring between the bar-type conductors and the tool (for example, setting crown) and damage being caused to the bar-type conductors or the insulation thereof, the stator or the tool (for example, the setting crown) is preferably caused to perform a follow-up movement in an axial direction. This movement follows a complex speed profile because the bar-type conductors are three-dimensionally deformed.

Preferably, radially adjacent bar-type conductors are twisted in opposite directions, such that at least two tools (for example, setting crowns) are used.

After the setting process, it is advantageous if all conductor ends have the same length in an axial direction (measured from the top edge of the laminated core). Owing to the different, radius-dependent deformation of the conductors during the setting process, it is preferable that the hairpins have a different length before the process. Accordingly, in the case of a housing of a stator before the flaring process, conductor ends preferably protrude to different lengths in a manner dependent on the radius—with increasing radius, the length in an axial direction also increases. In the case of a flared stator (the conductor ends have been radially shaped), the radial spacings between the wire ends have been increased, and the wire ends lie in predetermined tolerance ranges, such that the introduction into the pockets during an automated setting process is facilitated. A housing with wire ends that have been flared in a radial direction thus forms the preferred starting state for the setting process.

Devices and methods according to particularly preferred refinements of the invention offer, in particular, the advantages that a relative movement between the individual bar-type conductors and the setting tool can be prevented with a relatively simple mechanical structure in order to prevent damage to the conductors. For this purpose, it is preferable for each receiving and rotating unit (in particular, with setting crown) to follow a rotational and translational speed profile.

If all tools (for example, setting crowns) for the wire ends protruding with different radial positions were now to rotate at the same speed (and with the same starting time), a different axial stroke would be necessary, which would lead to a correspondingly complex mechanical structure for permitting the different axial stroke of the different tools.

In the case of preferred devices and methods according to the invention, different receiving and rotating units are moved with different speed profiles, such that it is merely necessary for the entire housing (for example, of a stator) to be axially moved—this is made possible with the individual angular speed of each receiving and rotating unit (for example, with setting crown).

Preferably, in order to prevent a relative movement and keep the number of drives low, the setting process is newly defined: the setting crowns or similar tools are moved only in rotation in order to make the twisting possible. The tools (for example, setting crowns) may be displaced into one another, such that the bending edge is situated at the same (axial) height in the case of all tools.

Preferably, the housing (for example, of a stator) is moved at a constant speed in an axial direction as far as the end position (after the setting process). In order to avoid the above-stated disadvantages, the tools (for example, setting crowns) are moved in rotation with an individual speed profile. In the case of an equal setting angle (twist angle), the



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conductors lying on a relatively large radius would, owing to their deformation, require a greater axial stroke than conductors on a relatively small radius. To adapt this, it is preferable if not the axial stroke for the different wire ends but rather the setting angle/twist angle is adapted.

In relation to the prior art, advantageous refinements of the invention make the individual movement of the individual tools (for example, setting crowns) possible with a small number of drives.

In the case of a preferred course of the relative axial movement between the housing and the bending device, the housing moves at a constant speed axially in the direction of the bending device (for example, with the setting crowns) proceeding from a time  $t_0$ . Preferably, the outermost receiving and rotating unit (for example, with setting crown) **K1** begins to rotate at a defined time  $t_1$ .

The outermost receiving and rotating unit **K1** furthermore preferably rotates with increasing speed in order to counterbalance the (axial) length compensation that arises as a result of the deformation, and in order to prevent a relative movement. As soon as the housing has reached a defined axial spacing, the closest receiving and rotating unit (for example, with a second setting crown) **K2** situated further to the inside furthermore preferably begins to twist the conductors. The second receiving and rotating unit rotates preferably in the opposite direction to the first receiving and rotating unit **K1**.

The second receiving and rotating unit preferably begins to rotate with a higher starting speed than the first receiving and rotating unit.

The speed of the second receiving and rotating unit **K2** preferably also increases.

In a preferred case in which the second receiving and rotating unit twists shorter wire ends than the first receiving and rotating unit, it is furthermore preferable that the speed of the second receiving and rotating unit **K2** is higher than that of the first receiving and rotating unit **K1**. It is thus possible that, even in the case of a later starting time, the same twist angle has been passed through at the end of the setting process ( $t_S$ ). The further receiving and rotating units with the further crowns **K3** and **K4** preferably behave in a manner equivalent to that which has been described on the basis of the movement of the first and second receiving and rotating units.

In a preferred construction of the bending device, which is designed, for example, as a setting device, each of the setting crowns is driven by means of an NC axis (for example electric lifting cylinder). For example, six setting crowns are used.

Particularly preferred refinements of the invention provide radial support of wire ends during the shaping in a circumferential direction. Advantageous features and advantages of such refinements will be discussed below.

Preferred refinements of the invention relate to a supplementation of the embodiments of a setting device discussed above. Radial support means, which are designed preferably as axially movable sleeves on the inner and outer sides, serve for the guidance of the hairpin in the setting process.

During the setting, if no inner radial support means—inner sleeve—is provided, there is the tendency for the wire ends to deform, as viewed in a radial direction, along the shortest path between stator and setting crown, that is to say, to change from the curved shape generated as a result of the preceding flaring to a substantially straight shape, owing to the tensile stress generated in the non-guided free wire ends as a result of the setting.

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To at least partially prevent this, in preferred embodiments, the undesired deformation of the wire ends in a radial direction during the setting is limited by the inner sleeve.

Furthermore, during the setting, the outer wire has a tendency to buckle where the wire end enters the cutout of the setting tool. This is prevented by means of an outer radial support means—in particular outer sleeve.

The radial support improves the setting result such that the risk of the occurrence of wire damage in the setting process is further minimized, and the specifications with regard to structural space limitations can be even better adhered to.

A particularly advantageous approach for the shaping of the wire ends in a circumferential direction (setting) will be described below:

Before the setting process, the pin ends are introduced into the setting tools.

Moving an inner and an outer guide sleeve (examples for an inner and an outer radial support means) out into the upper end position, that is to say, out of the setting device in the direction of the stator. In order to prevent a collision of the inner sleeve with the stator or the collar support fingers of the clamping device, said inner sleeve is particularly preferably of flexible design. For this purpose, in one preferred refinement, said inner sleeve is designed such that it moves no further once it has made abutting contact.

Setting process.

Moving the clamping device together with the stator upward to such an extent that the pin ends are (just) still positioned in the pockets of the setting tools (it is preferable for not the setting device, but rather only the stator and the clamping device, to be axially displaceable).

Moving the inner and the outer guide sleeve into the lower end position, that is to say, back into the setting device. (Comb-like) positioning fingers of a positioning device are moved between the wires in order to fix the wire ends after they have been pulled out of the setting tool, in order to enable said wire ends to be introduced, in the state thus fixed, into cutouts of a welding template.

Since the positioning fingers of the positioning device are radially advanced directly above the setting crowns, that is to say, in the vicinity of the wire ends, it is advantageous for the guide sleeves to be axially moved, that is to say, moved in, after the setting and before the radial advancing of the positioning fingers.

In the case of a process which does not require the positioning fingers, an axial movement (inward movement) of the guide sleeves is not imperatively necessary.

It is, however, preferable for the inner sleeve to be designed to be movable or at least flexible in order to avoid gaps between the sleeve and the stator or the clamping device. This does not apply to the outer sleeve, because its region of action lies in the immediate vicinity of the setting tool and therefore a gap can be present in the direction of the stator, or is even advantageous owing to the displacement of the stator during the setting process. The inner sleeve is moved with a predetermined pressure, for example of 3-4 bar, against the stop, which in refinements of the invention is formed, for example, by the interior of the clamping device.

If, during the setting process, the spacing between setting device and stator or clamping device is reduced, the inner sleeve preferably correspondingly moves conjointly. By contrast, the outer sleeve preferably has a gap in the direc-



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tion of the stator or clamping tool, and therefore does not need to be designed to be movable during the setting process.

A movable or flexible inner sleeve yields the following advantages:

despite the axially changing spacing of setting crowns and laminated core/clamping device during the setting process, the hairpins can be optimally guided at all times. This is preferably achieved by virtue of the inner sleeve being moved out and, during the setting process, being pushed axially downward by the clamping device with pressure reduction, that is to say being movable in a flexible manner, and not rigid.

For the purposes of automating a sub-process during the course of the production of a component, which is to be equipped with coils, of an electric machine, preferred refinements of the invention therefore provide a wire end shaping device for use during production of a component, which is equipped with coils, of an electric machine, for shaping wire ends which protrude from an annular housing of the component, comprising a bending device for bending wire ends in a circumferential direction, a relative movement device for moving the housing and the bending device in a relative manner in an axial direction, and a control unit. Here, the receiving and rotating units, which receive the wire ends, of the bending device are held so as to be axially static relative to one another, wherein differences in length compensation and/or in a turning angle of the wire ends are handled by means of different movement profiles of the rotational movements of the receiving and rotating units. A component production device and a wire end shaping method are also described.

According to a further aspect, the invention also provides a computer program product, with program commands which, when loaded onto a computer, in particular a control unit of a wire shaping device, activate the wire shaping device to carry out one of the methods according to the appended claims.

The device according to one or more of the appended device claims is designed for carrying out a method according to one or more of the appended method claims. The method according to one or more of the appended method claims can be performed by means of a device according to one or more of the appended device claims.

Although specific exemplary embodiments are described for the purposes of explaining the invention, further exemplary embodiments are disclosed to a person skilled in the art by this in that one or more of the described features may be omitted or replaced by alternative means.

## BRIEF DESCRIPTION OF THE DRAWINGS

Exemplary embodiments of the invention will be discussed in more detail below on the basis of the appended drawings, in which:

FIG. 1 is a schematic block illustration of an exemplary embodiment of a component production device for producing a component, which is equipped with coils, of an electric machine;

FIG. 2 is a schematic illustration of a housing of the component, which is equipped with coils, before a wire shaping process;

FIG. 3 shows a section through the housing as in FIG. 2 after a shaping of wire ends in a radial direction and before a shaping of the wire ends in a circumferential direction;

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FIG. 4 shows a plan view of a subregion of the housing from above in FIG. 2, wherein a groove of the housing with wire segments, on which wire ends are formed, accommodated therein is illustrated;

FIG. 5 is an illustration similar to FIG. 4, with a clamping device engaging on the wire ends for the purposes of clamping the latter;

FIG. 6 is a schematic overview illustration of an exemplary embodiment of a wire end shaping device, which has a radial wire end shaping device, for shaping the wire ends in a radial direction, with a radial bending device and the clamping device and has a circumferential wire end shaping device, for shaping the wire ends in a circumferential direction, with a circumferential bending device and the clamping device;

FIG. 7 is a perspective illustration of a first exemplary embodiment of the clamping device;

FIG. 8 shows an enlarged plan view of a detail of the clamping device from FIG. 7;

FIG. 9 shows a plan view of a second exemplary embodiment of the clamping device;

FIG. 10 shows a perspective view of the second exemplary embodiment of the clamping device;

FIG. 11 shows a section through the clamping device according to the second exemplary embodiment along the line A-A in FIG. 9;

FIG. 12 shows a perspective view of an exemplary embodiment of the radial bending device;

FIG. 13 shows a plan view of the exemplary embodiment of the radial bending device;

FIG. 14 is a perspective illustration of an exemplary embodiment of a gripper unit with a the radial bending device of FIGS. 12 and 13;

FIG. 15 is a perspective illustration of an exemplary embodiment of the circumferential bending device;

FIG. 16 shows a plan view of the circumferential bending device of FIG. 15;

FIG. 17 shows a diagram in which the relative speed between a housing and the circumferential bending device is plotted schematically versus the time;

FIG. 18 shows a diagram, recorded at the same time as FIG. 17, in which rotational speeds of a first to fourth receiving and rotating unit of the circumferential bending device are illustrated schematically in simplified form;

FIG. 19 is a perspective illustration of a fourth receiving and rotating unit of the circumferential bending device, wherein a ring element for forming wire end receptacles has been omitted for explanatory purposes;

FIG. 20 shows a side view of a first receiving and rotating unit;

FIG. 21 shows a section through the first receiving and rotating unit along the line B-B of FIG. 20;

FIG. 22 shows a perspective view of the first receiving and rotating unit;

FIG. 23 is an exploded illustration, in a side view, of the first receiving and rotating unit, wherein a main element and a ring element are illustrated in a state in which they are away from one another;

FIG. 24 is a perspective exploded illustration of the first receiving and rotating unit;

FIG. 25 is a perspective illustration of a combination of the first receiving and rotating unit, a second receiving and rotating unit, a third receiving and rotating unit and the fourth receiving and rotating unit nested one inside the other;



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FIG. 26 shows a side view of the combination of FIG. 25;

FIG. 27 shows a section through the combination along the line A-A of FIG. 26;

FIG. 28 shows a section through a combination of a first to sixth receiving and rotating means of the circumferential bending device of FIGS. 15 and 16, wherein each receiving and rotating means has a receiving and rotating unit and a drive element extending in a circumferential direction, in an operational position, with a section along the line B-B in FIG. 29 being illustrated;

FIG. 29 shows a plan view of the combination of FIG. 28;

FIG. 30 shows the combination similarly to FIGS. 28 and 29, wherein a first receiving and rotating unit has been removed, in section along the line C-C in FIG. 31;

FIG. 31 shows a plan view of the combination, with the first receiving and rotating unit having been removed, of FIG. 30;

FIG. 32 shows the combination of FIGS. 30 and 31 with the first receiving and rotating unit having been removed, but in an assembly and disassembly position for the assembly or disassembly of the second receiving and rotating unit, in section along the line D-D in FIG. 33;

FIG. 33 shows a plan view of the combination of FIG. 32 in the assembly and disassembly position;

FIGS. 34a and 34b are schematic diagrammatic illustrations for explanation of the function of a radial support means provided in a further embodiment of the circumferential bending device;

FIG. 35 shows a sectional view through the further embodiment of the circumferential bending device with a first and a second radial support means in a retracted position; and

FIG. 36 shows a view as in FIG. 36 with the first and the second radial support means in a supporting position.

#### DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

FIG. 1 illustrates an exemplary embodiment of a component production device 10 for producing a component, which is equipped with coils, of an electric machine in a block diagram, whereby production steps of a corresponding component production method for producing the component are also illustrated.

The component production device 10 is, in particular, suitable for producing a stator of an electric machine. The stator is configured to be used, in particular, as a stator of a traction motor of an electric vehicle, preferably in the power range from 20 kW to 400 kW. For this purpose, the stator should be provided with as large as possible a number of coils, wherein the coils may be produced from so-called hairpins 12.

In the illustrated embodiment, the component production device 10 has a wire end shaping device 14 and preferably one or more or all of the additional stations mentioned in more detail below.

The component production device 10 preferably has a housing production device 16 for producing a housing 18 of the component, which is designed and configured, for example, as a stator. The housing production device 16 is, for example, designed and configured in a fundamentally known manner to produce the housing 18 as a laminated core from individual laminations, wherein the housing 18 is of annular form and, on an inner wall region, is equipped with a series of housing grooves 20 which are formed so as to be distributed over the inner wall and which serve for receiving wire segments.

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The component production device 10 preferably has a device 22 for producing groove insulators, by means of which the individual housing grooves 20 are equipped with a groove insulator 24, preferably composed of insulating paper. The device for producing groove insulators 22 is preferably designed and configured in the manner described in more detail in the German patent application DE 10 2017 129 474.0, wherein, for further details, reference is expressly made to said German patent application DE 10 2017 129 474.0 which is incorporated herein by reference.

The component production device 10 furthermore has a hairpin production device 26 for producing the hairpins 12. The hairpin production device 26 may, for example, have cutting devices (not illustrated) for cutting wire pieces from a wire coil and bending devices (not illustrated in any more detail) for bending the hairpin 12 into a roof-shaped bend and/or kinked bend and/or 3-dimensional bend.

The component production device 10 preferably has a pre-positioning device 28 for pre-positioning the hairpin 12 and a hairpin insertion device 30 for inserting the thus pre-positioned hairpin 12 into the housing grooves 20 of the housing 18.

Possible embodiments of the pre-positioning device 28 and of the hairpin insertion device 30 are presented and described in more detail in the German patent application DE 10 2017 113 617.7, which is incorporated herein by reference for further details.

As indicated at the hairpin production device 26, the hairpins 12 have a bent winding head 32 and two free wire ends 34, wherein said hairpins are equipped with an insulator, for example a plastics coating, at each wire end 34, with the exception of the outermost end region. After the hairpin insertion by means of the hairpin insertion device 30, the wire ends 34 protrude from the housing grooves 20 at one end of the housing 18.

The wire end shaping device 14 serves for performing a flaring process, in which the individual wire ends 34 are flared in a radial direction, and for performing a setting process, wherein the thus flared wire ends 34 are shaped in a circumferential direction in order to thus each form pairs of wire ends 34 which are to be connected to one another.

For this purpose, the wire end shaping device 14 has a radial wire end shaping device 36 and a circumferential wire end shaping device 38.

The component production device 10 may furthermore have a preloading and/or fixing device 40 for preloading and/or fixing the individual pairs of wire ends 34. An exemplary embodiment of the preloading and fixing device 40 is presented and described in the German patent application DE 10 2017 114 932.5, which is incorporated herein by reference for further details.

The component production device 10 may furthermore comprise a wire end cutting device 42 for cutting the wire ends braced and fixed by means of the preloading and/or fixing device 40.

An exemplary embodiment of the component production device 10 furthermore has a wire end welding device 44 for welding the wire ends 34 which are to be connected to one another to form the coils.

Furthermore, the component production device 10 may comprise devices 46 for electrically contacting the coils thus formed by the hairpins 12 and/or for testing and/or potting the stator thus formed.

Exemplary embodiments of the wire end shaping device 14 of the component production device 10 will be discussed in more detail below.



In the case of the component production device 10 as per the exemplary embodiment illustrated in FIG. 1, for the production of a stator as an example for the component, a housing 18 with housing grooves 20 and groove insulators 24 accommodated therein is firstly provided; at the same time, conductors in the form of hairpins 12 are produced. The conductors in the form of hairpins 12 are subsequently fitted into the laminated core of the housing 18.

A wire end shaping method is subsequently carried out by means of the wire end shaping device 14. The initial state for the wire end shaping method is illustrated in FIG. 2. The sectional illustration in FIG. 2 shows an intermediate product for the production of the stator in a simplified illustration, in particular without the winding head being illustrated. In a first sub-process of the wire end shaping method, the conductor ends—wire ends 34—are radially shaped as illustrated in FIG. 3. This sub-process is also referred to as “flaring”. For this purpose, the wire end shaping device 14 has the radial wire end shaping device 36.

In this regard, FIG. 4 shows the plan view from above in FIG. 2 of one of the housing grooves 20. In the illustrated exemplary embodiment, a total of four wire segments 48a-48d with corresponding four wire ends 34a-34d protruding out of the housing groove 20 are illustrated. The wire end shaping method and the wire end shaping devices 14, 36, 38 provided for this are correspondingly designed and configured, in this example, with four wire ends 34a-34d per housing groove 20. In other embodiments which are not illustrated in any more detail, it is, for example, the case that 2, 3, 6, 8 or more or fewer wire ends 34 are provided per groove, wherein the number of corresponding handling elements, as will be presented in more detail further below, is correspondingly adapted.

For the automated processes that follow the flaring process, the wire ends 34a-34d should be shaped such that the position thereof lies within a predefined tolerance range. In order that the wire ends 34a-34d can be shaped in a reproducible manner, they are braced at their free end and at the foot—the upper edge of the housing 18.

For this purpose, a clamping device 50 is provided, exemplary embodiments of which are illustrated in FIGS. 5 and 7-11.

An exemplary embodiment of the wire end shaping device 14 is schematically illustrated in FIG. 6. Accordingly, an exemplary embodiment of the wire end shaping device 14 has the clamping device 50 and at least one bending device 52, 54.

The illustrated wire end shaping device 14 has at least two stations, a first station for the radial shaping of the wire ends 34a-34d and a second station for the shaping of the wire ends 34, 34a-34d in the circumferential direction. These two stations are referred to here as radial wire end shaping device 36 and circumferential wire end shaping device 38.

The radial wire end shaping device 36 has a radial bending device 52 for bending the wire ends 34, 34a-34d in the radial direction.

The circumferential wire end shaping device 38 has a circumferential bending device 54 for the shaping of the wire ends 34, 34a-34d, which have preferably already been radially flared, in the circumferential direction. Furthermore, both stations 36, 38 each have the clamping device 50.

Preferably, the wire end shaping devices 14, 36, 38 each have a relative movement device 56, 58 for moving the housing 18 relative to the respective bending device 52, 54 in an axial direction. In this regard, the “axial direction” is to be understood to mean a movement in a direction along the central axis of the annular housing 18. The radial

direction and the circumferential direction are also each defined in relation to this central axis of the housing 18.

A first relative movement device 56 is formed together with the radial bending device 52 and the clamping device 50 at the radial wire end shaping device 36. A second relative movement device 58 is formed together with the circumferential bending device 54 and the clamping device 50 at the circumferential wire end shaping device 38. Furthermore, the wire end shaping device 14 has a transport device 60 for transporting the housing 18 from the radial wire end shaping device 36 to the circumferential wire end shaping device 38 and a control device 62 for controlling the individual units of the wire end shaping device 14.

The transport device 60 may be designed and configured in any form. In the exemplary embodiment illustrated, the transport device 60 has a type of carousel with multiple arms, on which the relative movement devices 56, 58 are formed. The transport device 60 and the relative movement devices 56, 58 may be formed by a robot or a portal machine.

The relative movement devices 56, 58 each have a housing holder 64, on which the housing 18 is held with downwardly directed protruding wire ends 34, 34a-34d, and a housing holder drive 66, by means of which the housing holder 64 is movable in a driven manner relative to the respective bending device 52, 54 in an axial direction. In other refinements, which are presently less preferred, the housing 18 is held in a static manner, and the bending device 54 or 56 is moved axially.

The control device 62 has a first controller for controlling the radial wire end shaping device 36, which first controller activates the housing holder drive 66 of the first relative movement device 56 and the radial bending device 52. Furthermore, the control device 62 has a second controller 70 which activates the housing holder drive 66 of the second relative movement device 58 and the circumferential bending device 54. In the exemplary embodiment illustrated, the control device 62 has a third controller 72 which activates the transport device 60. The control device 62 may be provided locally at the wire end shaping device 14 or may also be a part of a central controller (not illustrated) of the component production device 10. The individual control devices 68, 70, 72 may be implemented in hardware or software form.

The clamping device 50 is preferably used both during the radial shaping of the wire ends in the radial wire end shaping device 36 and during the shaping of the wire ends 34, 34a-34d in the circumferential direction in the circumferential wire end shaping device 38. For this purpose, the clamping device 50 may be transported onward, together with the housing 18, from the radial wire end shaping device 36 to the circumferential wire end shaping device 38 by means of the transport device 60. The clamping device 50 may, for example, also be attached to, or be part of, the housing holder 64. In some embodiments, the clamping device 50 is provided as a separate component.

As illustrated in FIG. 6, the clamping device 50 is, during operation, arranged between the housing 18 and the respective bending device 52, 54.

As indicated in FIG. 5, the clamping device 50 has a first clamping means 74, which is preferably designed and configured as an inner clamping means, and a second clamping means 76, which is preferably designed and configured as an outer clamping means.

The first clamping means 74 is preferably arranged partially in the interior of the housing 18 and is furthermore preferably supported against inner wall regions 78 (see FIG.



5) which are formed, in the interior of the housing **18**, between the housing grooves **20**.

The second clamping means **76** is movable relative to the first clamping means **74** in order to clamp wire ends **34**, **34a-34d** between the clamping devices **74**, **76**. It is preferable here for all wire ends **34**, **34a-34d** which jointly protrude from a housing groove **20** to be clamped between the clamping devices **74**, **76**.

A first exemplary embodiment of the clamping device **50** will be discussed in more detail below on the basis of FIGS. **5**, **7** and **8**.

In this embodiment, the first clamping means **74** is of disk-shaped or annular form. The first clamping means **74** has an annular disk body **80** which is fitted with a first subregion into the housing and which is preferably supported against, or fixed relative to, the inner wall regions **78**. For this purpose, the annular disk body **80** may, for example, be axially braced in the annular housing **18** by means of a counter-bracing element (not illustrated in any more detail) on the opposite end of the housing. For example, the counter-bracing element may be formed as part of the housing holder **64**. A further subregion of the annular disk body **80** protrudes, as indicated in FIG. **5**, from the end of the housing **18** and is supported by way of an annular flange **82** against said end of the housing **18**. This protruding subregion of the annular disk body **80** is, as illustrated in FIGS. **7** and **8**, equipped with a series of cutouts **84** distributed over the circumference, which cutouts serve as tip receptacles **85**. The number of cutouts **84** corresponds to the number of housing grooves **20** on the housing **18**. The cutouts **84** are radially outwardly open.

The second clamping means **76** has a number of collar support tools in the form of clamping fingers **86**. Furthermore, the second clamping means **76** has, in the illustrated exemplary embodiment, a holding ring **87** on which the clamping fingers **86** are received so as to be guided in a jointly and synchronously driven and radially displaceable manner.

The clamping fingers **86** are each of bar-like form and are received with a region in the holding ring **87**. The clamping device may be designed and configured, for example, as a manual device without a drive. Here, the clamping fingers **86** are moved manually. In the embodiment illustrated in FIG. **7**, cylindrical pins **88** are provided for fixing purposes; it would also be possible for manual clamping to be performed by means of screws **94**. In other embodiments (not illustrated), that region of the clamping fingers **86** which is received in the holding ring **87** may in each case have a tooth rack region which meshes with toothed gears (not illustrated) which can be driven jointly by means of protruding toothed-gear shafts. In another embodiment which is likewise not illustrated in any more detail here, a corresponding drive cam is provided instead of a toothed gear.

Furthermore, the clamping fingers **86** may be equipped with spring elements which are not illustrated in any more detail here, for example in each case one pressure spring per clamping finger **86**, which spring elements ensure a uniform contact pressure of the clamping fingers **86**.

As can be seen most clearly from FIG. **8**, a respective rod element **90** of the clamping finger **86**, which is guided in the holding ring **87**, has a slot **92** which extends in the longitudinal direction of the clamping finger **86** and through which a fastening screw **94** is guided, wherein the shank of the fastening screw **94** limits the displacement movement of the clamping fingers **86** by way of the abutment in the slot **92**.

At the inwardly directed ends, the clamping fingers **86** have an engagement end in the form of a conically tapering tip **96**, which is designed and configured for engaging into the cutout **84**.

Furthermore, each clamping finger **86** has, at its free end, a step **98** which is delimited at one side by the conically tapering tip **96** and which is delimited at a side extending transversely with respect thereto by the rod element **90**, which is of thicker form.

That edge of the step **98** which is to be arranged so as to be averted from the housing **18** and which extends in the circumferential direction and which is formed on the rod element **90** is rounded in order to form a first bending formation **100**. Likewise, an edge which is to be arranged so as to be averted from the housing **18** and which extends in the radial direction is formed at the tip **96** in order to form a bending formation **102**, **104** for the setting process. This applies both to the edge in the region of the step **98** and to the edge on the other side of the tip **96**, which thus likewise forms a rounded bending formation **104** for the setting process.

In the clamping position illustrated in FIGS. **7** and **8**, the tips **96** have been introduced into the tip receptacles **85** on the annular flange **82**; the steps **98** and the tips **96** form, with the tips **96** of the adjacent clamping fingers **86**, receiving grooves **106** for a group of wire ends **34**, **34a-34d** which protrude together from a housing groove **20**. This group of wire ends **34a-34d** is clamped between a first clamping region **108** on the first clamping means **74** and a second clamping region **110** on the second clamping means **76** when the clamping fingers **86** are moved radially inward. The first clamping region **108** is formed on the annular flange **82** in each case between the tip receptacles **85**. The second clamping region **110** is formed on that region of the step **98** which is delimited by the rod element **90**. Those edges which are averted from the housing **18** and which are formed around the respective receiving groove **106** thus act as bending formations **100**, **102**, **104** for the wire shaping. During the wire shaping, the clamping fingers **86** are supported at one side in the holding ring **87** and at the other side by the receiving of the tips **96** in the tip receptacle **85**.

The second embodiment of the clamping device **50** will be discussed in more detail below on the basis of FIGS. **9-11**. The second embodiment corresponds to the first embodiment aside from the differences presented in more detail below, such that reference may be made to the description above, wherein the same reference designations have also been used for corresponding elements.

The second embodiment may be used, in particular, where little space is available for the insertion and fastening of the first clamping means **74**.

For this purpose, instead of the annular disk body **80**, an arrangement of several radially movable segments **112** has been provided, which are guided movably in a radial direction between a main body **114** and an annular disk **116** and which can be driven by means of a conical drive element **120**, which can be moved axially by means of a drive thread **118**. The upper edge, illustrated in FIG. **2**, of the housing **18** with the wire ends **34**, **34a-d** can be introduced into an annular depression **122** which is formed between the radially movable segments **112** and the holding ring **87**, wherein, then, the radially movable segments **112** can, by axial displacement of the conical drive element **120**, be moved radially outward in order to thus be moved against the inner wall regions **78** of the housing **18**, such that the clamping device **50** is thus fixed to said inner wall regions **78**.



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In both of the embodiments of the clamping devices **50** illustrated here, the wire ends **34a-34d** then protrude from each housing groove **20** beyond the clamping device **50**, as indicated in FIG. **6**, and can be shaped by the respective bending device **52**, **54**.

An exemplary embodiment of the radial bending device **52** will be discussed in more detail below on the basis of the illustrations in FIGS. **12-14**.

The embodiment of the radial bending device **52** illustrated in FIGS. **12-14** has a base plate **124**, which is rotatable about an axis of rotation, a base plate drive means **126**, which can be activated by the first controller **68** and which serves for driving the rotational movement of the base plate **124**, a drive plate **128**, which is rotatable relative to the base plate **124** likewise about the axis of rotation, a drive plate drive means **130**, which serves for driving the rotational movement of the drive plate **128** and which can be activated by the first controller **68**, and a series of grippers **132**, which are arranged so as to be distributed over a circumference. In each case one gripper **132** is formed on a gripper unit **134**, which is illustrated individually in FIG. **14**.

For this purpose, on the base plate **124**, there are provided displacement guides, for example in the form of a displacement rail **136**, which displacement guides extend in a radial direction. In each case one gripper carriage **138** is guided displaceably on the displacement guide. On the gripper carriage **138**, a first gripping jaw **140** and a second gripping jaw **142** are guided so as to be displaceable relative to one another in a tangential direction. For example, a guide block **144** with a first guide groove **146** for guiding the gripping jaw **140** and a second guide groove **148** for guiding the second gripping jaw **142** are formed on the gripper carriage **138**. Furthermore, each gripper unit **134** has a jaw drive unit which is not illustrated in any more detail in the drawings. The jaw drive units of all gripping jaws may be driveable synchronously by means of a common drive means which is likewise not illustrated in any more detail. Alternatively, the jaws of all, a group of or individual grippers **132** may be driveable individually, for example by means of electric motors. The jaw drive means or the jaw drive units are likewise activatable by means of the first controller **68**.

As also emerges from FIG. **14** and FIGS. **12** and **13**, each gripper carriage **138** has a slotted-guide pin **150** which engages into a corresponding slotted guide **152**, provided for each gripper **132**, in the drive plate **128**. In the illustrated exemplary embodiment, the slotted guide **152** is formed in each case by a slot extending obliquely with respect to the radial direction. A relative rotation between drive plate **128** and base plate **124** thus leads, by means of the slotted guide **152** formed as a slot, to a displacement of the gripper carriages **138** and thus of the grippers **132** along the displacement rail **136** in a radial direction. In this way, all grippers **132** are movable jointly in the radial direction.

As can be seen most clearly from FIG. **14**, the individual grippers **132** are designed and configured in the form of pincers. For this purpose, on each gripping jaw **140**, **142**, there is provided in each case one free end **154** which projects inward in a radial direction, wherein, at the tip of the free end **154**, there is provided a projection **156** which projects in the direction of the other gripping jaw. Those surfaces of the projections **156** which are directed toward one another form the clamping surfaces **158** for the clamping of the wire ends **34a-34d** which are to be bent in each case. For this purpose, the dimensions of the clamping surfaces **158** are selected such that always in each case only one wire end **34a-34d** is clamped between the projections **156**. In some embodiments, clamping surfaces may be

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provided between two projections, such that the wire ends can be gripped in frictionally locking fashion at the clamping surfaces and in positively locking fashion at between the projections.

As can be seen from FIG. **6**, the first control device **68** furthermore also activates the first relative movement device **56**; this is in each case that relative movement device which has been moved by the transport device **60** into a position close to the radial bending device **52**.

That part of the wire end shaping method which leads to the flaring of the wire ends (the bending of the wire ends **34a-34d** from the position illustrated in FIG. **2** into the position illustrated in FIG. **3**) will be discussed in more detail below.

Firstly, the housing **18** passing from the hairpin insertion device **30** is fixed, by means of the wire ends **34** protruding from the housing grooves **20**, on the housing holder **64**, and the clamping device **50** is fixed to the housing **18** such that the clamping device **50** is arranged between the housing **18** and the radial bending device **52**. More specifically, the clamping device **50** is arranged at the end of the housing **18**, wherein the wire ends **34a-34d** which jointly protrude from a housing groove **20** are clamped, in each case, jointly between one of the clamping fingers **86** and the first clamping means **74**. The corresponding receiving groove **106** of the clamping device **50** is, in this case, designed and configured such that the groove insulator **24** which protrudes with its end remains protected.

Then, the housing **18** is, by means of the first relative movement device **56**, moved with the downwardly projecting protruding wire ends **34a-34d** axially in the direction of the radial bending device **52**.

Initially, by corresponding relative adjustment between base plate **124** and drive plate **128**, all of the grippers **132** have been moved into the position in which they have been moved furthest to the outside. The wire ends **34**, **34a-34d** which protrude further downward beyond the clamping device **50** are moved into the space radially within the grippers **132**. The base plate **124** is positioned such that each gripper **132** is aligned centrally with respect to a housing groove **20**. In the embodiment illustrated, fewer grippers **132** are provided than there are housing grooves **20**. For example, 64 housing grooves are provided, but only 8 grippers. Preferably, for the number  $n_{GN}$  of housing grooves and the number  $n_{Gr}$  of grippers **132**, the following applies:  $n_{GN} = M \times n_{Gr}$ , where  $M$ ,  $n_{GN}$  and  $n_{Gr}$  are natural numbers.

Thus, firstly, the wire ends **34a-34d** protruding from a subgroup of the housing grooves **20** are radially flared.

For this purpose, the first controller **68** activates the drive plate drive means **130** such that the grippers **132** are displaced, with gripper jaws **140**, **142** moved apart from one another, to the height of the first wire ends **34a** arranged radially furthest to the outside. Subsequently, the jaw drive means is activated in order to move the gripper jaws **140**, **142** toward one another and thus grasp the first wire ends **34a** by means of the grippers **132**. With the gripper jaws **140** moved together, the drive plate drive means **130** is then driven by the first controller **68** such that the grippers **132** move radially outward along the displacement rail **136** to a predetermined extent in order to thus bend the first wire ends **34a** radially outward. Here, the first rounded bending formation **100** gives rise to reproducible bending of the first wire ends **34a**. At the same time, the first relative movement device **56** is activated in order to compensate the relative position between housing **18** and radial bending device **52** in



accordance with the change, which can be set during the bending process, in the extent of the first wire ends **34** in an axial direction.

Subsequently, the gripping jaws **140**, **142** are moved apart, and the grippers **132** are moved, by activation of the drive plate drive means **130**, to the radial position of the second wire segments **48b**, while, at the start of the bending of the second wire ends **34b**, the axial relative position is set to a value specific for the second wire end. The second wire ends **34b** are then correspondingly clamped and/or held in positively locking fashion by virtue of the gripping jaws **140** being moved together, and are bent outward to a determined, smaller extent than the first wire ends **34a** by virtue of the grippers **132** being moved radially outward. Here, too, it is in turn the case that the axial relative position is readjusted in accordance with the change in the axial extent of the second wire ends **34b**.

This process is then repeated for the third wire end **34c**. In one embodiment, the fourth wire end **34d** remains in the original position. In another embodiment, the fourth wire end **34d** is also bent outward to a small extent in order to also ensure positioning of the fourth wire end in a narrow tolerance range. Bending inward, at least of some wire ends, for example of the innermost wire ends **34d**, is however basically also possible.

Subsequently, the grippers **132** move back into the radially outermost position, and the base plate drive means **126** is subsequently driven by the first controller **68** in order to rotate the base plate **124** to a certain extent until the grippers **132** have been set to the center of the next adjacent housing groove **20**. Here, the process of the flaring is then repeated again starting from the first, radially outermost wire end **34a** to the third wire end **34c** or possibly to the fourth wire end **34d**.

In the embodiment illustrated here, where the number nGr of grippers **132** is one eighth of the number nGN of housing grooves **20**, the flaring process is ended after an eightfold adjustment of the base plate **124**.

In other embodiments, not four but some other number of wire ends **34a-34d** is provided per housing groove **20**. In these embodiments, too, the flaring is performed analogously to the example discussed on the basis of the example with four wire ends **34a-34d** per housing groove **20**.

Possible embodiments of the circumferential bending device **54** will be discussed in more detail on the basis of the illustrations in FIGS. **15-36**.

Here, FIGS. **15** and **16** illustrate an overview of an embodiment of the circumferential bending device **54**. This embodiment has a first to sixth receiving and rotating means **160a-160f**. Each receiving and rotating means **160a-160f** has, in each case, one receiving and rotating unit **162a-162f**, for receiving a group of wire ends **34a** or **34b** or **34c** or **34d** with equal radial spacing to the housing center and for turning these wire ends **34a-34b**, **34c** or **34d** for the purposes of bending in a circumferential direction, and one drive means **164a-164f** for driving a rotational movement of the receiving and rotating unit **162a-162f**.

Each drive means **164a-164f** has an annular drive element **166a-166f**, which is mounted so as to be rotatable about the common axis of rotation, and an actuator **168a-168f** for driving the drive element **166a-166f** in rotation. In the exemplary embodiment illustrated, the drive elements **166a-166f** are designed and configured as annular disk elements with an arm, wherein the actuators **168a-168f** are designed and configured as linear drives, for example in the form of electric spindle drives.

Furthermore, each drive element **166a-166f** is rotatably mounted on an annular support disk **170a-170f** fastened in a static manner. The actuators **168a-168f** are mounted on machine frames **172** so as to be static relative to the annular support disks **170a-170f**. The annular support disks **170a-170f** are fastened jointly to fastening columns **174**. Thus, the drive elements **166a-166f**, and the receiving and rotating units **162a-162f** fastened thereto in a manner to be discussed in more detail below, are duly also rotatable relative to one another about a common axis of rotation, but are held so as to be static relative to one another in an axial direction. Only a rotary drive (first to sixth drive means **164a-164f**), but no separate axial drive, is necessary.

FIGS. **20** to **24** show the first receiving and rotating unit **162a** in different illustrations. FIG. **19** is a perspective illustration of the fourth receiving and rotating means **162d**, and FIGS. **25** to **27** show a combination of the first to fourth receiving and rotating units **162a-162d** stacked one inside the other. As can be seen from these illustrations, each receiving and rotating unit **162a-162f** has an annular arrangement of upwardly open receiving holes **176** which extend downward in an axial direction and which serves for receiving wire ends. The annular arrangement of receiving holes **176** is formed on setting crowns **180a-180f** which are arranged so as to lie around one another with different diameters at the same axial height, and which all end upwardly at the same axial height. Here, the first receiving and rotating unit **162a** has an annular arrangement of receiving holes **176** with the largest diameter. This is followed, as viewed in the inward direction, by the annular arrangement of the second receiving and rotating unit **162b**, then the annular arrangement of the third receiving and rotating unit **162c** etc., as far as the sixth receiving and rotating unit **162f**, the annular arrangement of which is arranged furthest to the inside.

Thus, the first receiving and rotating unit **162a** is designed and configured for receiving the first wire ends **34a** arranged radially furthest to the outside, wherein all first wire ends **34a** of all housing grooves **20** can be grasped and can be bent in the circumferential direction by rotation of the first receiving and rotating unit **162a**. The second receiving and rotating unit **162b** is designed and configured for receiving all second wire ends **34b** which protrude from all housing grooves **20**. For the wire end shaping process, the second receiving and rotating unit **162b** is turned in the opposite direction to the first receiving and rotating unit **162a** until such time as the first wire ends **34a** from one of the next housing grooves **20** come to lie at the same circumferential position as some of the second wire ends **34b**, such that pairs of wire ends **34a**, **34b** which are to be connected to one another come to lie adjacent to one another. The third receiving and rotating unit **162c** serves for receiving and bending the third wire ends **34c** in the circumferential direction, and the fourth receiving and rotating unit **162d** serves for receiving and bending the fourth wire ends **34d**.

The circumferential bending device **54** illustrated here also has a fifth and a sixth receiving and rotating unit **162e** and **162f**, such that fifth and sixth wire ends could also be bent. To produce the stator of which a precursor is illustrated in FIGS. **2** and **3**, where four wire ends **34a-34d** protrude from each housing groove **20**, the first to fourth receiving and rotating means **160a-160d** are sufficient, such that the fifth and sixth receiving and rotating means **160e**, **160f** could also be omitted. In the case of a correspondingly smaller or greater number of wire ends, the number of receiving and rotating means would need to be correspondingly adapted.



The bending of the wire ends **34a-34d** in the circumferential direction is performed, in each case, with opposite directions of rotation for the wire ends **34a/34b**, **34b/34c**, **34c/34d** which are arranged adjacent to one another in a housing groove **20**, specifically to such an extent that, with an adjacent wire end from one of the next housing grooves **20**, in each case one pair for connection can be produced such that multiple continuous coil windings are realized. Here, the, in each case, outer wire ends **34a** must, owing to the greater circumference at the outer side, be bent over a greater distance than the wire ends **34d** arranged, in each case, right at the inside.

Therefore, the hairpins **12** are preferably already produced and/or inserted into the housing **18** such that, as illustrated in FIG. **2** and FIG. **3**, the first wire ends **34a** protrude with a greater length than the second wire ends **34b**, the second wire ends **34b** protrude with a greater length than the third wire ends **34c**, and the third wire ends **34c** in turn protrude with a greater length than the fourth wire ends **34d**. The differences in the lengths are dependent, in each case, on the differences in the distance to be covered in the circumferential direction.

During the bending of the wire ends **34a-34d** by rotation of the corresponding receiving and rotating unit **162a-162d**, the respective axial extent of the wire end **34a-34d**, which is bent progressively further, shortens owing to the bending process.

Therefore, during the setting process in the circumferential wire end shaping device **38**, the second relative movement device **58** is correspondingly readjusted in order to compensate the length compensation. The corresponding activation is performed by means of the second controller **70**, which both activates all drive means **164a-164f**, and thus, via the actuators **168a-168f**, the drive elements **166a-166f**, and also activates the second relative movement device **58**.

Here, there are different resulting movement profiles **K1**, **K2**, **K3**, **K4** for the wire ends **34a-34d** of different length and the different distances of the bending in the circumferential direction for the respective wire ends **34a-34d** with different radial spacing to the housing central axis.

Instead of now providing a separate movement in an axial direction for each receiving and rotating means **160a-160f** for the different length compensation, the rotational movement of the individual receiving and rotating units **162a-162f** is controlled differently such that, in a manner dependent on the relative movement between housing **18** and circumferential bending device **54**, and in a manner dependent on the axial change in length resulting from the bending of the respective wire ends **34a-34d** in the circumferential direction, the rotational movement of each receiving and rotating unit **162a-162f** is individually activated such that, in the receiving holes **176**, there is no resulting relative movement between the respective end region of the wire end **34a-34d** and the receiving hole **176**.

Greatly simplified examples for corresponding movement profiles **K1-K4** are illustrated in FIGS. **17** and **18**. Here, FIG. **17** shows the movement profile of the second relative movement device **58**, more specifically the relative speed  $v$  between the housing **18** and the circumferential bending device **54**. FIG. **18** shows, at the same time, the rotational speeds  $\omega$  for the first to fourth receiving and rotating unit **162a-162d**.

As can be seen from FIG. **17**, it is preferably the case here that the housing **18** is moved with a uniform speed  $v$  toward the circumferential bending device **54**.

The first wire ends **34a** are, after the previously discussed flaring process, situated at the radial height of the annular arrangement of receiving holes **176** of the first receiving and rotating unit **162a**, and are, shortly before the time  $t_1$ , moved with their end regions to a certain extent into the receiving holes **176**. Here, the time  $t_1$  is selected such that an adequately long end region of the first wire ends **34a** has been moved into the receiving holes **176** of the first receiving and rotating unit **162a**. This is followed, as illustrated in the curve **K1**, by the start of a rotational movement of the first receiving and rotating unit **162a** in the first direction of rotation (positive axis of FIG. **18**). The rotational speed  $\omega$  is in this case not uniform but rather varies correspondingly to the requirement arising from the uniform speed of the housing **18** toward the circumferential bending device **54** and from the axial change in length of the first wire ends **34a** that results during the rotation. The curve **K1** is depicted here in simplified form as linearly rising. The actual course of the movement is somewhat more complex, but can be easily ascertained by means of simple tests, or mathematically. The target criterion here is that the end regions do not move in an axial direction within the receiving holes.

Shortly before the time  $t_2$ , the end regions of the second wire ends **34b** move into the receiving holes **176** of the second receiving and rotating unit **162b** until, at the time  $t_2$ , said end regions have been moved in to a sufficient extent. At this time  $t_2$ , the rotation of the second receiving and rotating unit **162b** in the opposite direction of rotation to the rotation of the first receiving and rotating unit **162a** (negative region in FIG. **18**) begins. The corresponding rotation of the second receiving and rotating unit **162b** is likewise depicted in simplified form as linearly rising by the curve **K2**. Here, too, the actual course of the movement is more complex. The curves **K3** and **K4**, with their start points  $t_3$  and  $t_4$ , show the respective corresponding movement profile for the third receiving and rotating unit **162c** and the fourth receiving and rotating unit **162d**, respectively. Here, if a stator with six wire ends per housing groove **20** is processed, corresponding curves would also be depicted for the fifth and sixth receiving and rotating units **162e**, **162f**.

It is clear from FIGS. **17** and **18** that the different receiving and rotating units **162a-162f** are driven with different movement profiles in order, with a simple mechanical construction without separate axial drives, to effect the length compensation during the shaping of the wire ends **34a-34d** in the circumferential direction, such that as far as possible, no relative movement of the end regions of the wire ends **34a-34d** in the respective wire end receptacle **182** formed by the receiving holes **176** occurs.

During the bending of the wire ends **34a-34d** in the circumferential direction by means of the circumferential bending device **54**, as discussed above, the clamping device **50** is furthermore arranged between the housing **18** and the bending device **54**. The second rounded bending formation **102** on the clamping fingers **86** acts here as a bending formation for those wire ends **34a-34c** which are to be bent in one direction of rotation, and the third rounded bending formation **104** on the clamping fingers **86** acts here as a bending formation for those wire ends **34b**, **34d** which are to be shaped in the other direction of rotation. Owing to the clamping device **50**, the position of the wire ends **34a-34d** is also maintained in a defined manner during the shaping of the wire ends **34a-34d** in the circumferential direction. The shaping is performed in a highly reproducible manner. Furthermore, the groove insulators **24** are protected.

After the completion of the wire end shaping process (setting), the clamping device **50** is removed again, for



which purpose, firstly, the clamping fingers **86** are moved radially outward and thus moved out of the intermediate spaces between the wire ends **34a-34d**; subsequently, the clamping device **50** can be released from the housing **18**, and removed from the housing **18** in an axial direction.

The housing with the shaped wire ends **34a-34d** is then transported onward to the next station in the component production device **10**, for example to a preloading and/or fixing device **40**, where the pairs of wire ends are captured and/or braced together for the purposes of carrying out cutting and welding processes.

As discussed above, for the shaping of the wire ends **34a-34d**, the end regions thereof are pushed axially into an annular arrangement of receiving holes **176** and then set by turning of the annular arrangement.

The receiving holes **176** are preferably formed on so-called setting crowns **180a-180f** which, in the case of automated mass production of stators, are correspondingly highly loaded and should therefore be serviced and/or exchanged from time to time.

The particular configuration of preferred exemplary embodiments for the circumferential bending device **54**, which permit easy exchange and/or easy assembly and disassembly of the receiving and rotating units **162a-162f**, will be discussed below on the basis of the illustrations in FIGS. **19** to **33**.

FIG. **19** shows an example for the fourth receiving and rotating unit **162d**, and FIGS. **20** to **24** show an example for the first receiving and rotating unit **162a**. As can be seen from FIGS. **19** to **24**, the receiving and rotating units **162a-162f** have the setting crowns **180a-180f**, on which are formed the annular arrangement of receiving holes **176**, which act as wire end receptacles **182**. The setting crowns **180a-180f** are provided at one end of a cylinder-shell-shaped region **184a-184f**, at the other end of which there is formed an annular disk flange **186a-186f**, by means of which the receiving and rotating unit **162a** is connected by means of a releasable connection **188a-188f** to the respective drive element **166a-166f** formed as a drive ring.

FIGS. **25** to **27** illustrate a combination of the first to fourth receiving and rotating units **162a-162d** nested one inside the other, whereas FIGS. **28-33** show a combination of the first to sixth receiving and rotating means **160a-160f** in different positions.

Here, the first to sixth drive elements **166a-166f** are arranged one above the other, wherein the first drive element **166a** is arranged at the very top, and the sixth drive element **166f** is arranged at the very bottom, and the remaining drive elements **166b-166e** are arranged in a distributed manner in between. Correspondingly, the cylinder-shell-shaped regions **184a-184f** have different extents in an axial direction, wherein the first cylinder-shell-shaped region **184a** is that region with the shortest axial extent, and the sixth cylinder-shell-shaped region **184f** is that which has the longest axial extent. The sixth cylinder-shell-shaped region has the smallest outer circumference, which is dimensioned so as to just fit into the inner circumference of the fifth cylinder-shell-shaped region **184e**. The fourth cylinder-shell-shaped region **184d** is arranged around this; the first cylinder-shell-shaped region **184a** is arranged at the very outside; the remaining cylinder-shell-shaped regions **184c** are arranged in between.

As can be seen most clearly from FIGS. **28** to **33**, each of the first to sixth releasable connections **188a-188f** has a series of connecting bridges **190a-190f** arranged so as to be distributed over the circumference. Each connecting bridge **190a-190f** extends radially and has, in each case, one

output-side bridge element **192a-192f** on the receiving and rotating unit **162a-162f**, a drive-side bridge element **194a-194f** on the drive element **166a-166f**, and a connecting element **196a-196f** for the releasable connection of the bridge elements **192a-f**, **194a-f**.

As can be seen most clearly from FIGS. **19** to **24**, the output-side bridge elements **192a-192f** are formed as radially extending projections which are distributed over the circumference of the annular flange **186a-186f** and which are in the form of screw-attachment flanges with bores which extend in an axial direction over one half of the axial of the annular flange **186a-186f**.

The drive-side elements **194a-194f** are formed as projections which protrude inwardly at the inner circumference of the annular drive elements **196a-196f** and which are in the form of screw-attachment flanges with threaded bores, the axial height of which corresponds to the second half of the axial extent of the annular flange **186a-186f**. The connecting elements **196a-196f** are formed predominantly by threaded screws, which can be screwed with the threaded shank thereof into the threaded bores of the drive-side bridge elements **194a-194f**, wherein the screw heads of said threaded screws bear against the rims of the bores of the output-side bridge elements **192a** and can be engaged from above by means of a tool engagement portion. One of the connecting elements, for example that which is in the region of an arm, is designed and configured differently, for example is formed by a projection on one of the bridge elements **192a-f**, **194a-f** and a complementary set-back portion on the other of the bridge elements, in order to thus ensure a predetermined relative position.

As shown by a comparison of FIGS. **19** to **24** and the illustrations in FIGS. **25** to **27**, the output-side bridge elements **192a-192f** which are distributed over the circumference of the annular flange **196a-196f** form a toothed structure, the toothed pitch of which is the same on the first to sixth receiving and rotating units **162a-162f**. Between the connecting bridges **190a-190f**, there are formed passages **198a-198f**, the clear width of which in a circumferential direction is dimensioned to be greater than the maximum extent of the bridge elements **192a-f**, **194a-f** in the circumferential direction. It is thus possible, if the receiving and rotating unit **162a-f**, on the one hand, and drive element **166a-166f**, on the other hand, have been correspondingly turned relative to one another, for the bridge elements **192a-192f**, **194a-f** to be guided past one another in the axial direction.

As can be seen from FIGS. **19** to **24**, the setting crowns **180a-180f** are each formed by virtue of the axially extending longitudinal grooves **200** being milled in at the end of the cylinder-shell-shaped region **184a-184f**, wherein said longitudinal grooves **200** have insertion bevels **202** at their free ends. The number of longitudinal grooves **200** corresponds to the number  $n_{GN}$  of housing grooves **20**. A setting crown ring **204** is pushed onto this arrangement of longitudinal grooves **200** distributed over the circumference, such that the receiving holes **176** are delimited at three sides by the longitudinal groove edges and at the radially outer side by the setting crown ring **204**.

To exchange these setting crowns **180a-180f**, a disassembly and assembly method is performed which will be discussed in more detail below on the basis of the illustrations in FIGS. **28** to **33**.

Here, FIGS. **28** and **29** show an initial position and an operational position of the circumferential bending device **54**. Here, the first connecting elements **196a** are firstly released. This can be performed easily by engagement of a



tool on the screw heads. Subsequently, the first receiving and rotating unit **162a** can be removed in an upward direction, which leads to the situation illustrated in FIGS. **30** and **31**. As can be seen from FIG. **31**, all of the bridge elements **192a-192f**, **194a-194f** are in this case aligned relative to one another.

The second controller **70** includes a disassembly mode and assembly mode, by means of which the individual drive elements **166a-166f** are individually movable for movement into a respective disassembly position.

This is illustrated in FIGS. **32** and **33** for the first drive element **166a**. This first drive element **166a** is, in the disassembly position, turned relative to the second drive element **166b**, and the second receiving and rotating unit **162b** that is still fastened thereto, such that the second connecting bridges **190b** align with the first passages **198a**; preferably, the centers of the second connecting bridges **190b** are aligned with the centers of the first passages **198a**. In this position, the screw heads of the second connecting elements **196b** are accessible through the first passages **198a**, such that the second connecting elements **196b** can be removed.

After the removal of the second connecting elements **196b**, the second receiving and rotating means **160b** can be removed by displacement in an upward axial direction, wherein the second output-side bridge elements **192b** can be guided through the first passages **198a** between the first drive-side bridge elements **194a** of the first drive element **160a**. Subsequently, the second drive element **166b** is also moved into the disassembly position, wherein the first and the second drive-side bridge elements **194a**, **194b** align with one another and the third connecting elements **196c** are accessible through the first and second passages **198a**, **198b**. After removal of the third connecting elements, the third receiving and rotating unit **162c** can be removed upward in an axial direction, wherein the third output-side bridge elements **192c** are guided through the corresponding first and second passages **198a**, **198b** between the corresponding drive-side bridge elements **194a**, **194b** of the first and second drive element **166a**, **166b**. This process can be repeated for the removal of the further receiving and rotating means **160d-160f**.

The assembly of the new receiving and rotating means **160f-160a** is then performed correspondingly in the reverse sequence, wherein the process is commenced with the sixth receiving and rotating means **160f** while the first to fifth drive elements **166a-e** are in the disassembly position.

A yet further embodiment of the circumferential wire end shaping device **38** will be discussed below on the basis of FIGS. **34a**, **34b**, **35** and **36**. This embodiment of the circumferential wire end shaping device **38** corresponds substantially to the embodiment of the circumferential wire end shaping device **38** discussed above, and has the same features, such that, in this regard, reference is made to the description above. Additionally, this embodiment of the circumferential wire end shaping device **38** also has at least one radial support means **210** for radially supporting wire ends **34** during the shaping of the wire ends **34** in the circumferential direction.

The mode of operation of such a radial support means **210** will be discussed in more detail in FIGS. **34a** and **34b**. Here, FIG. **34a** shows the housing **18** of the component with a wire end **34** protruding therefrom, which wire end is to be shaped in the circumferential direction by the circumferential wire end shaping device **38** and, for this purpose, has been introduced into one of the receiving and rotating units **162a-162f**, wherein, as illustrated in FIG. **34b**, the wire end

**34** bears against the respective bending formation **102**, **104** of the clamping finger **86** of the clamping device **50**.

If the shaping, also referred to here as setting, of the wire end **34** in the circumferential direction is now performed, then the wire end **34** has the tendency, indicated by dotted lines in FIG. **34a**, to extend rectilinearly over the shortest path, such that the shaping in the radial direction (flaring) that has been performed by the radial wire end shaping device **36** would be subjected to shaping again.

To prevent this, the circumferential bending device **54** has at least one radial support means **210** which radially supports the wire end during the setting, such that the shape in the radial direction, shown by solid lines, is maintained.

In particular, the embodiment of the circumferential bending device **54** shown in FIGS. **35** and **36** has an inner radial support means **210a**, formed as an inner sleeve **212**, and an outer radial support means **210b**, formed as an outer sleeve **214**. The radial support means **210**, **210a**, **210b** are axially movable between a retracted position illustrated in FIG. **35** and a supporting position illustrated in FIG. **36**.

The sleeves **212**, **214** are formed as annular sleeves with a circular circumference and an axis of symmetry which coincides with the axis of rotation of the receiving and rotating units **162a-162f**. The sleeves **212**, **214** are movable between the retracted position and the supporting position, for example hydraulically or pneumatically, and in a manner controlled by the second controller **70**. The movement from the retracted position into the supporting position is performed with a predetermined maximum pressure, for example of 3 to 4 bar, such that the movement is stopped when the maximum pressure is reached in the event of impacting against an abutment or against an obstruction. As a result, at least the inner sleeve **212** is of flexible design.

The outer radial support means **210b** formed as an outer sleeve **214** lies, in the supporting position, radially at the outside against wire ends **34**, **34a**, such that outward bulging of the wire ends **34**, **34b** is prevented.

The radial support means **210**, **210a**, **210b** are controlled such that, firstly, the retracted position illustrated in FIG. **35** is set when the wire ends are moved into the receiving holes **176**. The supporting position of the radial support means **210**, **210a**, **210b** shown in FIG. **36** is thereupon assumed before the rotational movement of the receiving and rotating units **162a-162f** begins. The rotational movement and thus the shaping of the wire ends **34** in the circumferential direction is performed with the radial support means **210**, **210a**, **210b** having been moved in this supporting position. Here, the control pressure on the inner sleeve **212** is reduced, such that the inner sleeve can move conjointly, correspondingly to the desired axial movement of the wire ends **34**, during the shaping.

Subsequently, the radial support means **210**, **210a**, **210b** are in turn moved into the retracted position shown in FIG. **35**, and the preloading and/or fixing device **40** can be engaged for the purposes of preloading and/or fixing the wire ends **34**. The component is then, together with this preloading and/or fixing device **40**, moved out of the engagement with the circumferential bending device **54**, by virtue of the wire ends **34** being pulled out of the receiving holes **176**. The component is then transported onward to the next processing step of the component production device **10**.

While at least one exemplary embodiment of the present invention(s) is disclosed herein, it should be understood that modifications, substitutions and alternatives may be apparent to one of ordinary skill in the art and can be made without departing from the scope of this disclosure. This disclosure is intended to cover any adaptations or variations



of the exemplary embodiment(s). In addition, in this disclosure, the terms “comprise” or “comprising” do not exclude other elements or steps, the terms “a” or “one” do not exclude a plural number, and the term “or” means either or both. Furthermore, characteristics or steps which have been described may also be used in combination with other characteristics or steps and in any order unless the disclosure or context suggests otherwise. This disclosure hereby incorporates by reference the complete disclosure of any patent or application from which it claims benefit or priority.

## LIST OF REFERENCE DESIGNATIONS

10 Component production device  
 12 Hairpin  
 14 Wire end shaping device  
 16 Housing production device  
 18 Housing  
 20 Housing groove  
 22 Device for producing groove insulators  
 24 Groove insulator  
 26 Hairpin production device  
 28 Pre-positioning device  
 30 Hairpin insertion device  
 32 Winding head  
 34 Wire end  
 34a First wire end  
 34b Second wire end  
 34c Third wire end  
 34d Fourth wire end  
 36 Radial wire end shaping device  
 38 Circumferential wire end shaping device  
 40 Preloading and/or fixing device  
 42 Wire end cutting device  
 44 Wire end welding device  
 46 Devices for electrical contacting and/or testing and/or potting, . . . .  
 48a First wire segment  
 48b Second wire segment  
 48c Third wire segment  
 48d Fourth wire segment  
 50 Clamping device  
 52 Radial bending device  
 54 Circumferential bending device  
 56 First relative movement device  
 58 Second relative movement device  
 60 Transport device  
 62 Control device  
 64 Housing holder  
 66 Housing holder drive  
 68 First controller  
 70 Second controller  
 72 Third controller  
 74 First clamping means (inner clamping means)  
 76 Second clamping means (outer clamping means)  
 78 Inner wall region  
 80 Annular disk body  
 82 Annular flange  
 84 Cutout  
 85 Tip receptacle  
 86 Clamping finger  
 87 Holding ring  
 88 Toothed-gear shaft  
 90 Rod element  
 92 Slot  
 94 Fastening screw  
 96 Tip

98 Step  
 100 First rounded bending formation (flaring)  
 102 Second rounded bending formation (setting)  
 104 Third rounded bending formation (setting)  
 5 106 Receiving groove of the clamping device  
 108 First clamping region  
 110 Second clamping region  
 112 Radially movable segment  
 114 Main body of the clamping device  
 10 116 Annular disk  
 118 Drive thread  
 120 Conical drive element  
 122 Annular depression  
 124 Base plate  
 15 126 Base plate drive means  
 128 Drive plate  
 130 Drive plate drive means  
 132 Gripper  
 134 Gripper unit  
 20 136 Displacement rail  
 138 Gripper carriage  
 140 First gripping jaw  
 142 Second gripping jaw  
 144 Guide block  
 25 146 First guide groove  
 148 Second guide groove  
 150 Slotted-guide pin  
 152 Slotted guide  
 154 Free end of a gripping jaw  
 30 156 Projection on the free end of the gripping jaw  
 158 Clamping surface  
 160a First receiving and rotating means  
 160b Second receiving and rotating means  
 160c Third receiving and rotating means  
 35 160d Fourth receiving and rotating means  
 160e Fifth receiving and rotating means  
 160f Sixth receiving and rotating means  
 162a First receiving and rotating unit  
 162b Second receiving and rotating unit  
 40 162c Third receiving and rotating unit  
 162d Fourth receiving and rotating unit  
 162e Fifth receiving and rotating unit  
 162f Sixth receiving and rotating unit  
 164a First drive means  
 45 164b Second drive means  
 164c Third drive means  
 164d Fourth drive means  
 164e Fifth drive means  
 164f Sixth drive means  
 50 166a First drive element  
 166b Second drive element  
 166c Third drive element  
 166d Fourth drive element  
 166e Fifth drive element  
 55 166f Sixth drive element  
 168a First actuator  
 168b Second actuator  
 168c Third actuator  
 168d Fourth actuator  
 60 168e Fifth actuator  
 168f Sixth actuator  
 170a First annular support disk  
 170b Second annular support disk  
 170c Third annular support disk  
 65 170d Fourth annular support disk  
 170e Fifth annular support disk  
 170f Sixth annular support disk



172 Machine frame  
 174 Fastening column  
 176 Receiving holes  
 180a First setting crown  
 180b Second setting crown  
 180c Third setting crown  
 180d Fourth setting crown  
 180e Fifth setting crown  
 180f Sixth setting crown  
 182 Wire end receptacle  
 184a First cylinder-shell-shaped region  
 184b Second cylinder-shell-shaped region  
 184c Third cylinder-shell-shaped region  
 184d Fourth cylinder-shell-shaped region  
 184e Fifth cylinder-shell-shaped region  
 184f Sixth cylinder-shell-shaped region  
 186a First annular flange  
 186b Second annular flange  
 186c Third annular flange  
 186d Fourth annular flange  
 186e Fifth annular flange  
 186f Sixth annular flange  
 188a First releasable connection  
 188b Second releasable connection  
 188c Third releasable connection  
 188d Fourth releasable connection  
 188e Fifth releasable connection  
 188f Sixth releasable connection  
 190a First connecting bridge  
 190b Second connecting bridge  
 190c Third connecting bridge  
 190d Fourth connecting bridge  
 190e Fifth connecting bridge  
 190f Sixth connecting bridge  
 192a First output-side bridge element  
 192b Second output-side bridge element  
 192c Third output-side bridge element  
 192d Fourth output-side bridge element  
 192e Fifth output-side bridge element  
 192f Sixth output-side bridge element  
 194a First drive-side bridge element  
 194b Second drive-side bridge element  
 194c Third drive-side bridge element  
 194d Fourth drive-side bridge element  
 194e Fifth drive-side bridge element  
 194f Sixth drive-side bridge element  
 196a First connecting element  
 196b Second connecting element  
 196c Third connecting element  
 196d Fourth connecting element  
 196e Fifth connecting element  
 196f Sixth connecting element  
 198a First passage  
 198b Second passage  
 198c Third passage  
 198d Fourth passage  
 198e Fifth passage  
 198f Sixth passage  
 200 Longitudinal groove  
 202 Insertion bevel  
 204 Setting crown ring  
 210 Radial support means  
 210a Inner radial support means  
 210b Outer radial support means  
 212 Inner sleeve  
 214 Outer sleeve

The invention claimed is:

1. A wire end shaping method for shaping wire ends during a course of production of a component of an electric machine, wherein the wire ends protrude from an annular housing, wherein the component is equipped with coils, the method comprising:

a) providing or using a bending device which has a first receiving and rotating unit, rotatable about an axis of rotation, for receiving first wire ends and which has a second receiving and rotating unit, rotatable about an axis of rotation, for receiving second wire ends which are radially offset with respect to the first wire ends, wherein the receiving and rotating units are rotatable relative to one another but are static or held relative to one another in an axial direction,

b) moving the housing and the bending device in an axial relative movement in order to introduce the first wire ends into the first receiving and rotating unit and the second wire ends into the second receiving and rotating unit and in order to compensate an axial change in length of wire ends during a bending operation,

c) rotating the first receiving and rotating unit relative to the housing to bend the first wire ends after end regions of the first wire ends have been introduced into the first receiving and rotating unit, and rotating the second receiving and rotating unit relative to the housing to bend the second wire ends after end regions of the second wire ends have been introduced into the second receiving and rotating unit,

wherein the rotation of the first and second receiving and rotating units is performed with different movement profiles in a manner dependent on the axial relative movement and on an axial length of the received wire ends such that further axial relative movement between the end regions of the wire ends and the respective receiving and rotating unit is prevented.

2. The wire end shaping method as claimed in claim 1, wherein at least one of:

the axial relative movement in step b) is performed with a uniform speed and the rotation in step c) is performed in each case with a varying rotational speed;

in step a), the bending device with a third receiving and rotating unit for receiving third wire ends, which are radially offset relative to the first and second wire ends, and for turning the third wire ends for bending in the circumferential direction is provided or used, wherein the third receiving and rotating unit is rotatable relative to the first and second receiving and rotating unit about the axis of rotation but is axially static or held and, in step c), after the introduction of end regions of the third wire ends, is rotated such that a relative movement of the end regions in the third receiving and rotating unit is prevented; or

in step a), the bending device with a fourth receiving and rotating unit for receiving fourth wire ends, which are radially offset relative to the first to third wire ends, and for turning the fourth wire ends for bending in the circumferential direction is provided or used, wherein the fourth receiving and rotating unit is rotatable relative to the first to third receiving and rotating unit about the axis of rotation but is axially static or held and, in step c), after the introduction of end regions of the fourth wire ends, is rotated such that a relative movement of the end regions in the fourth receiving and rotating unit is prevented.



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3. The wire end shaping method as claimed in claim 2, wherein at least one of:

rotational movements of individual receiving and rotating units are correspondingly, in a case of different lengths of the wire ends to be bent in each case, started at correspondingly different points in time, or

the rotational movements of all receiving and rotating units and a relative movement between housing and bending device are ended at the same time.

4. The wire end shaping method as claimed in claim 1, wherein the wire ends are fixedly clamped by means of a clamping device arranged between the housing and the bending device, during a bending process.

5. The wire end shaping method as claimed in claim 4, further comprising at least one of the following steps:

fixing a first clamping means to the housing or in a positionally static manner, and moving a second clamping means relative to the first clamping means to clamp the wire ends between the first and the second clamping means;

jointly clamping all wire ends which protrude out of a groove of a housing;

supporting a first of multiple clamping means, which are movable relative to one another, against inner wall regions of the housing, which inner wall regions are arranged between grooves of the housing;

radially moving multiple segments of a first of multiple clamping means, which are movable relative to one another, to at least one of fix the first clamping means to the housing or for release from the housing;

driving, by means of an axially movable drive element, a radial movement of multiple segments of a first of multiple clamping means, which are movable relative to one another, in order to at least one of fix the first clamping means to the housing or for release from the housing, and jointly transmitting the axial movement to the segments by means of a control surface or a conical surface;

abutting a first clamping region, which is formed on a first of multiple clamping means, which are movable relative to one another, against radially inner sides of wire ends, arranged radially at the inside, of the wire ends protruding out of the grooves of the housing;

radially moving a number of clamping fingers, provided correspondingly to a number of at least one of groups of wire ends to be clamped together or of grooves in the housing, to clamp the wire ends or for releasing the clamping;

clamping, in each case, one group of the wire ends between a first clamping means and, in each case, one radially movable clamping finger of a second clamping means;

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supporting clamping fingers, which are moving into a clamping position, against displacement in a circumferential direction or in an axial direction;

clamping fingers, which are moving into a clamping position, against displacement in a circumferential direction or in an axial direction by means of positively locking engagement of each clamping finger into one or on a complementary grasping unit;

supporting clamping fingers, which are moving into the clamping position, against displacement in a circumferential direction or in an axial direction by means of positively locking engagement of a tip of each clamping finger of a second clamping means into a corresponding tip receptacle on the first clamping means;

deforming the wire ends by abutment against bending formations or bending edges arranged on the clamping device;

at least one of jointly or synchronously driving a radial movement of clamping fingers to clamp the wire ends;

jointly guiding the radial movement of the clamping fingers on a holding ring; or

receiving the two, three, four, five, six or more wire ends, which are to be clamped and which jointly emerge from a groove of the housing, in a receiving groove which is formed by adjacent clamping fingers at free ends thereof, wherein the receiving groove is delimited, at one side running substantially in a radial direction and at one side running substantially in a circumferential direction, by one clamping finger, and at another side running substantially in a radial direction, by another clamping finger, and is open at another side running substantially in a circumferential direction.

6. The wire end shaping method as claimed in claim 1, further comprising the following step:

radially bending one or more wire ends, which jointly protrude from a groove of the housing, to flare said wire ends prior to an introduction into the bending device.

7. The wire end shaping method as claimed in claim 1, wherein step c) comprises:

c1) radially supporting at least some of the wire ends on at least one of a radially inner side or a radially outer side during a bending process.

8. The wire end shaping method as claimed in claim 7, wherein step c1) comprises:

axially moving at least one radial support means out from a retracted position into a supporting position before the bending process and axially moving the at least one radial support means in from the supporting position into a retracted position after the bending process.

\* \* \* \* \*