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(54) **RING GEAR MOUNTING ARRANGEMENT WITH OIL SCAVENGE SCHEME**

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(57) **ABSTRACT**

(51) **Int. Cl.**
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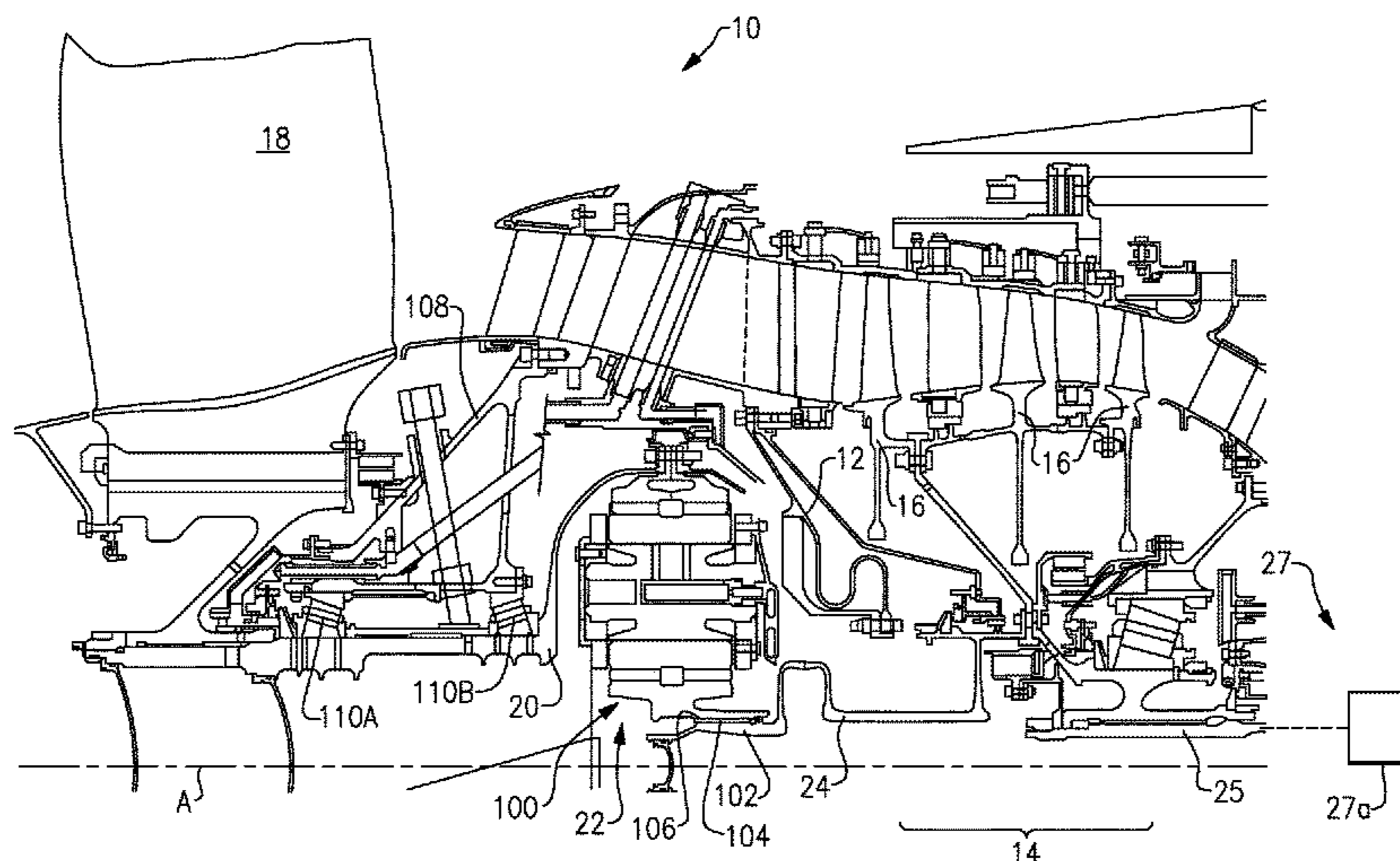
A gas turbine engine includes a bypass ratio greater than about ten (10). A fan is supported on a fan shaft and has a plurality of fan blades. A gear system is connected to the fan shaft. A plurality of planetary gears and a first set of opposed angled ring gear teeth are separated from a second set of opposed angled ring gear teeth. A lubricant flow path is located axially between the first set of opposed angled ring gear teeth and the second set of opposed angled ring gear teeth. A gear system support is relative to a fixed housing facilitating segregation of vibrations. An annular channel is axially aligned with the lubricant flow path. A low pressure turbine has an inlet, an outlet, and a low pressure turbine pressure ratio greater than 5:1.

(52) **U.S. Cl.**
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CPC ... F02K 3/06; F02C 7/36; F16H 57/04; F16H 57/0479

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30 Claims, 3 Drawing Sheets



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continuation of application No. 16/122,236, filed on Sep. 5, 2018, now Pat. No. 10,527,151, which is a continuation of application No. 15/892,210, filed on Feb. 8, 2018, now Pat. No. 10,082,105, which is a continuation of application No. 14/753,048, filed on Jun. 29, 2015, now Pat. No. 10,107,231, which is a continuation-in-part of application No. 13/346,120, filed on Jan. 9, 2012, now abandoned, which is a continuation-in-part of application No. 11/504,220, filed on Aug. 15, 2006, now Pat. No. 8,753,243.

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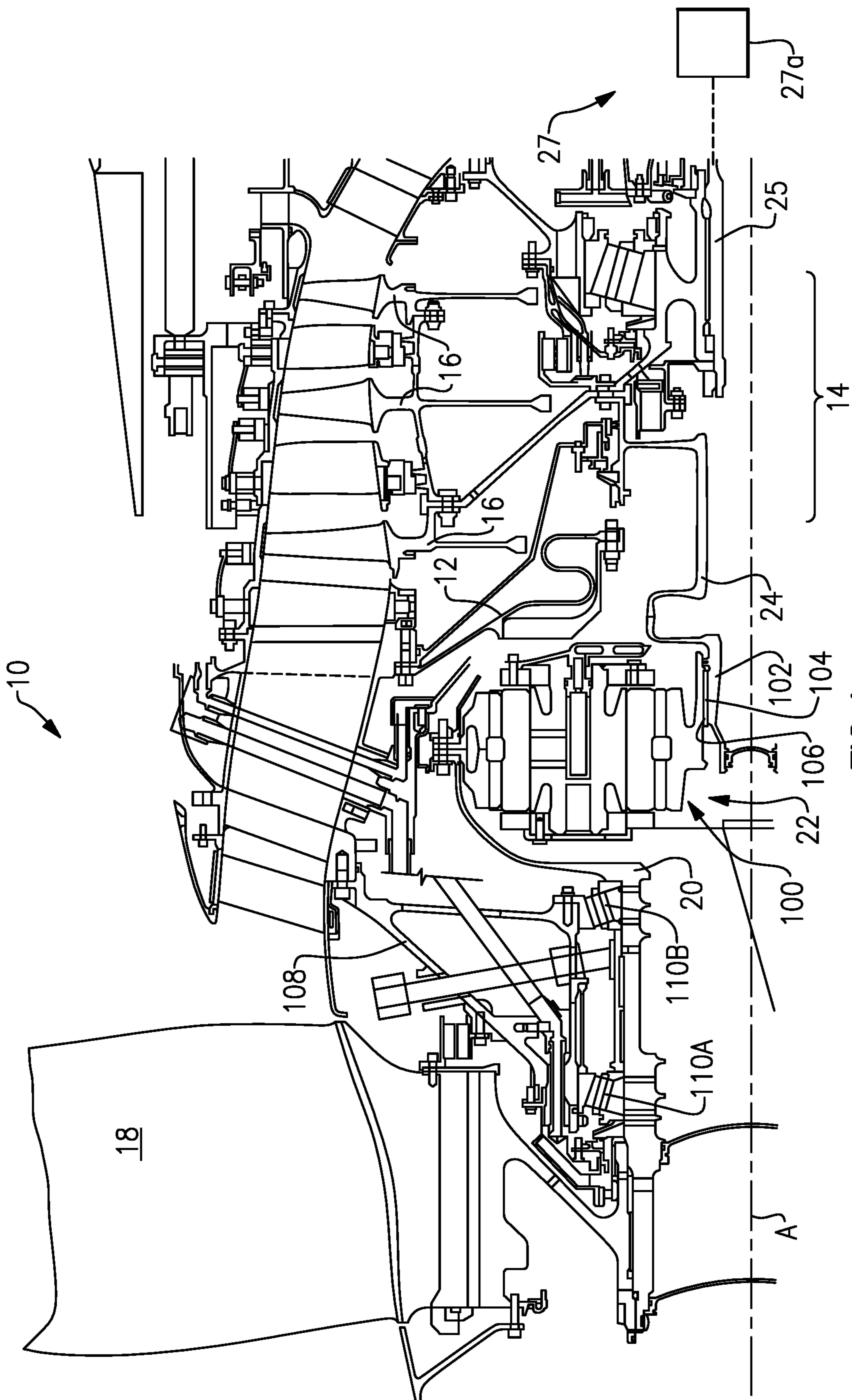


FIG. 1

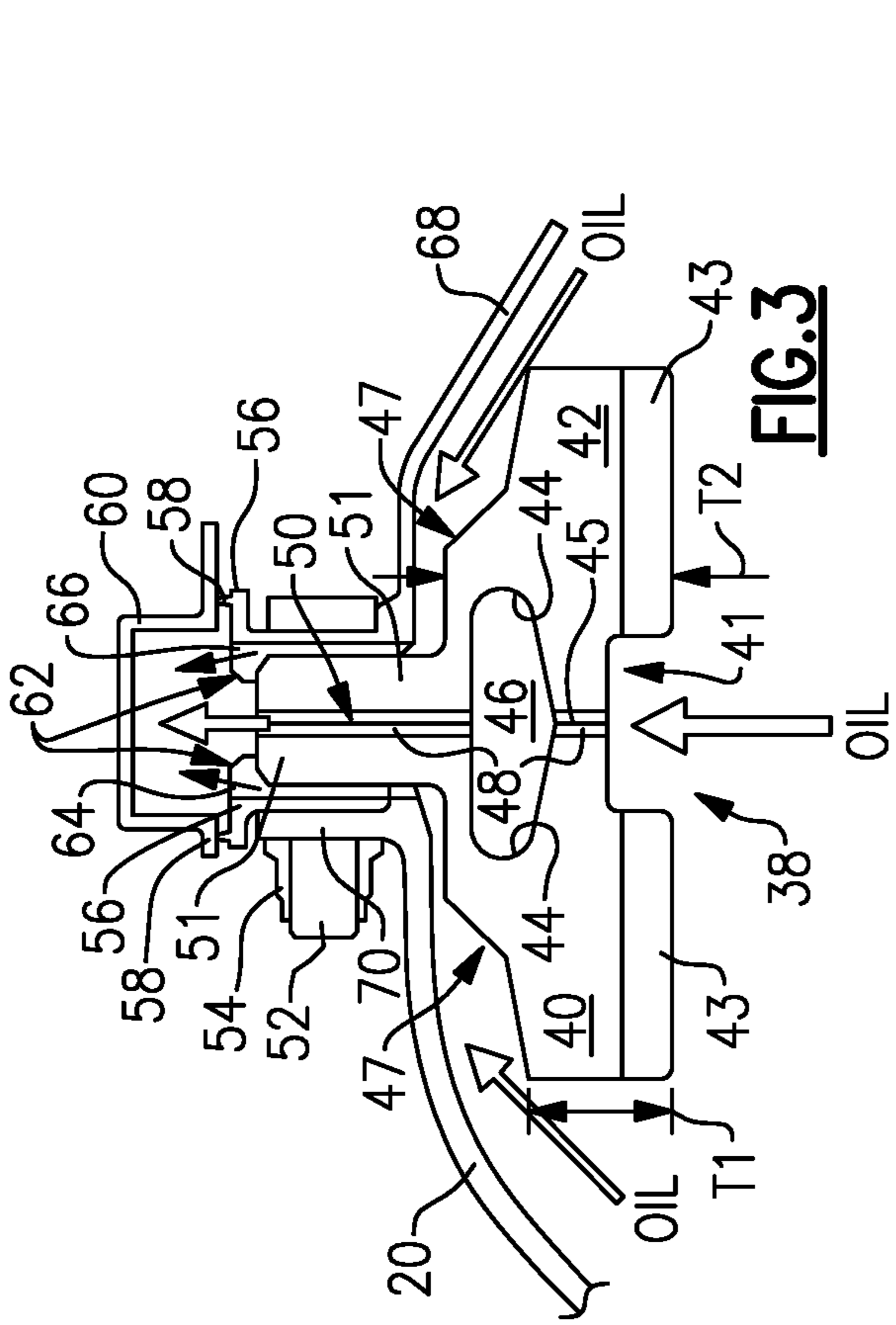


FIG. 3

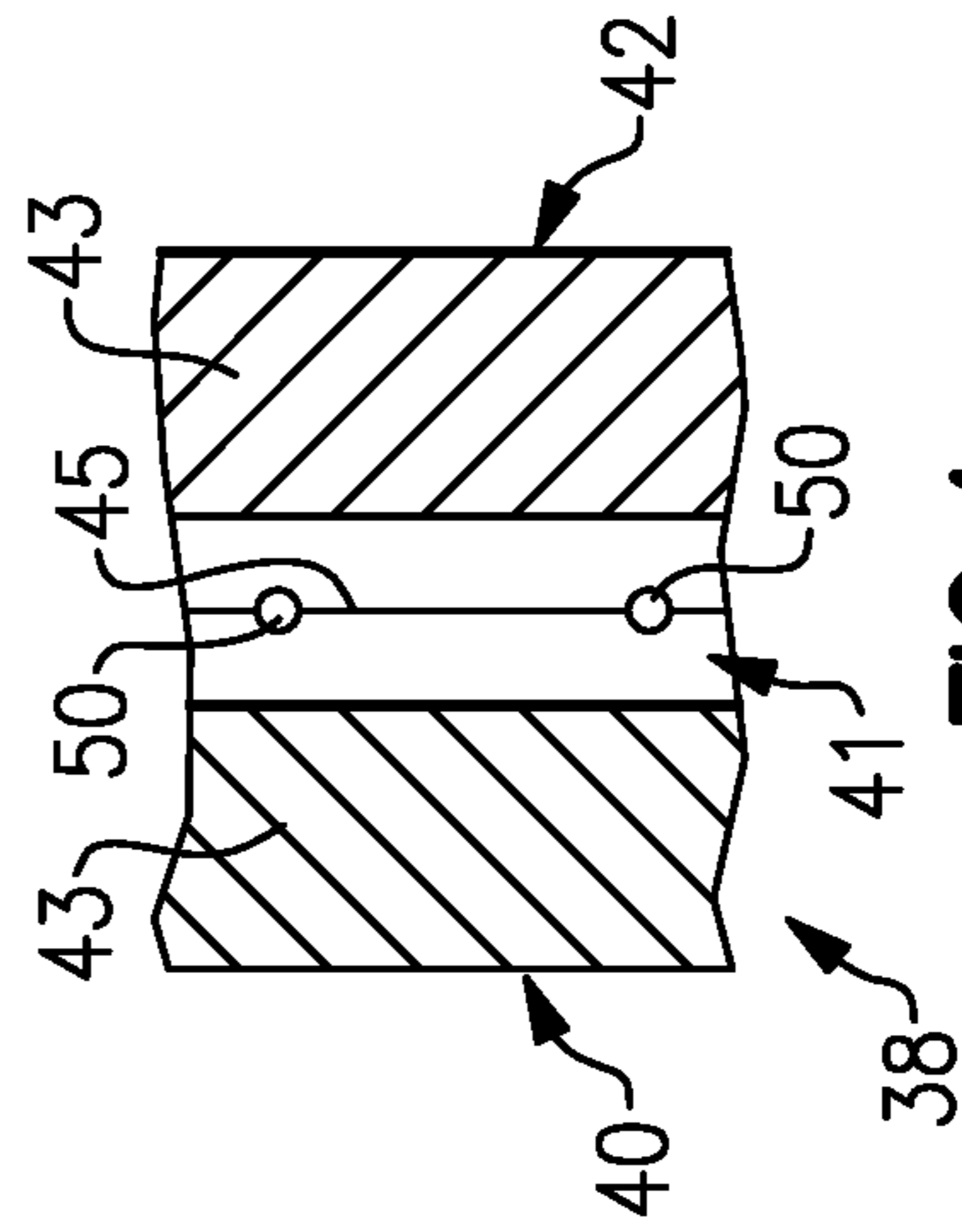


FIG. 4

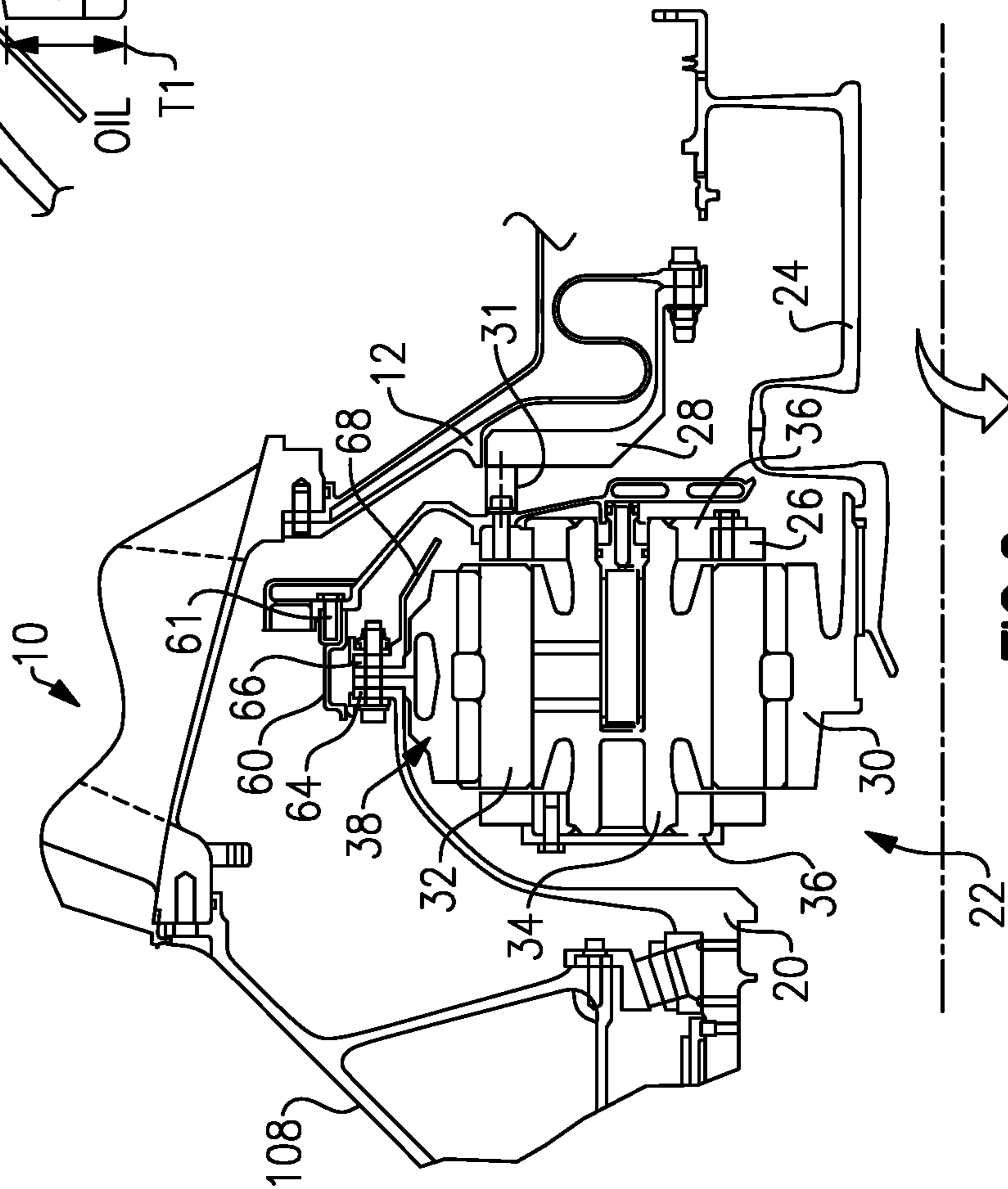


FIG. 2

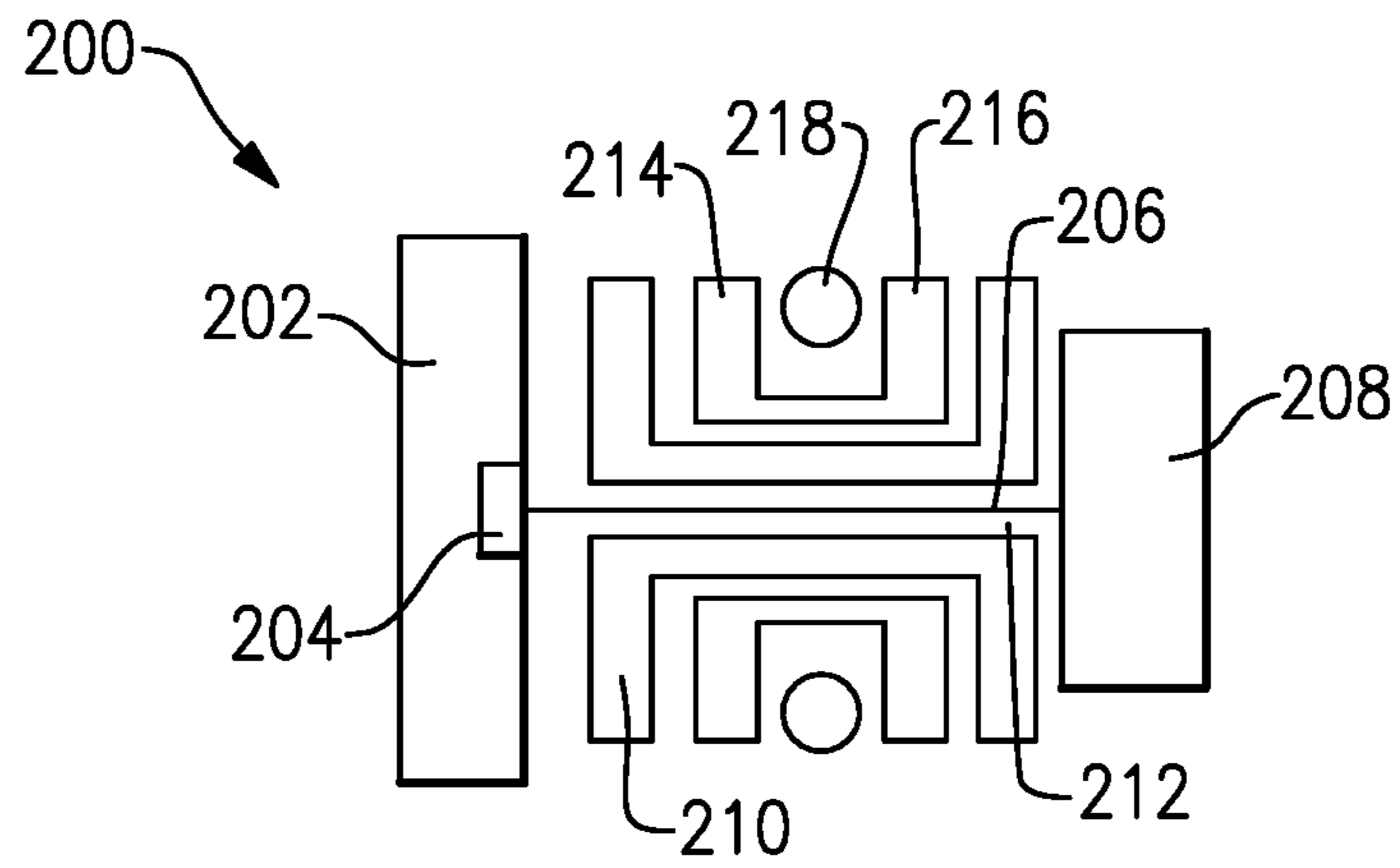


FIG. 5

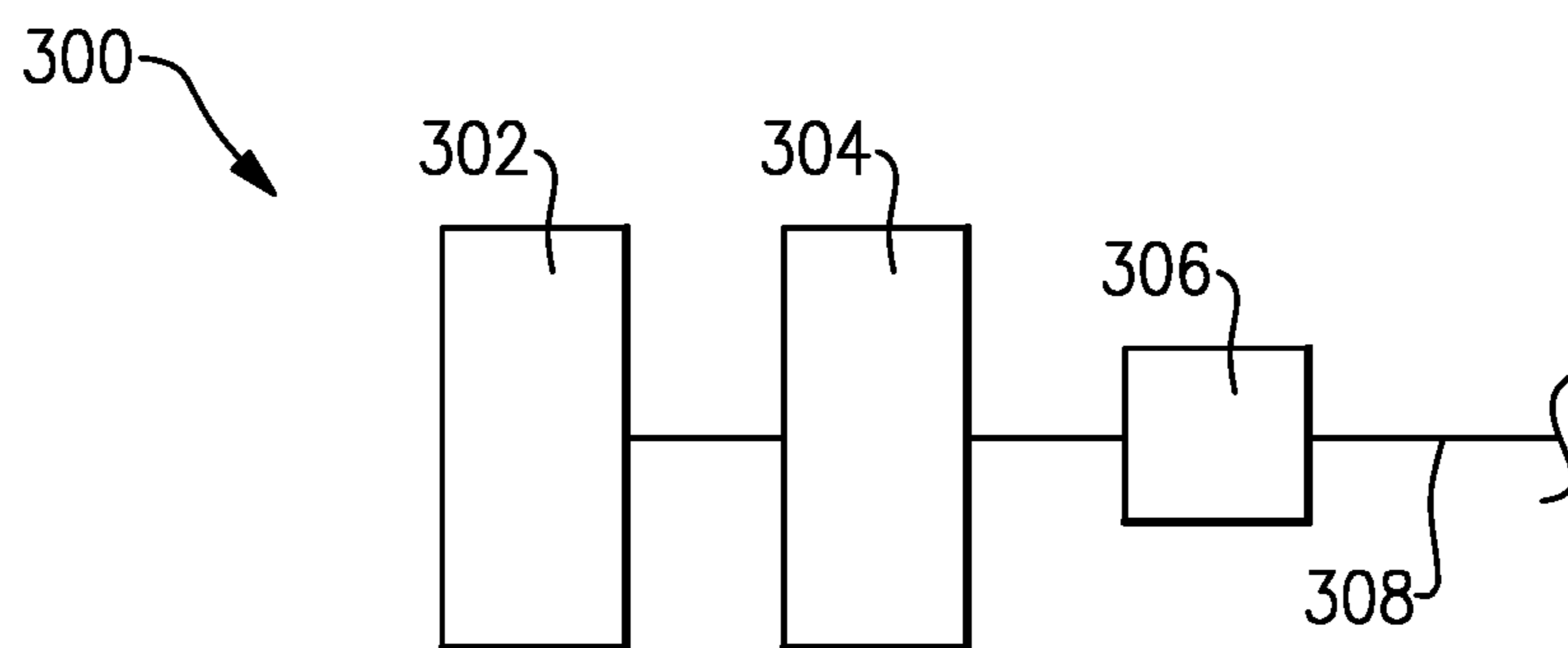


FIG. 6

RING GEAR MOUNTING ARRANGEMENT WITH OIL SCAVENGE SCHEME

REFERENCE TO RELATED APPLICATIONS

This application is a continuation of U.S. patent application Ser. No. 16/736,013 filed Jan. 7, 2020, which is a continuation of U.S. patent application Ser. No. 16/122,236 filed on Sep. 5, 2018, which is now U.S. Pat. No. 10,527,151 granted Jan. 7, 2020, which is a continuation of U.S. patent application Ser. No. 15/892,210 filed on Feb. 8, 2018, which is now U.S. Pat. No. 10,082,105, granted Sep. 25, 2018, which is a continuation of U.S. patent application Ser. No. 14/753,048 filed on Jun. 29, 2015, which is now U.S. Pat. No. 10,107,231, granted Oct. 23, 2018, which is a continuation-in-part of U.S. patent application Ser. No. 13/346,120, filed on Jan. 9, 2012, which is a continuation-in-part of U.S. patent application Ser. No. 11/504,220, filed on Aug. 15, 2006, which is now U.S. Pat. No. 8,753,243 granted Jun. 17, 2014.

BACKGROUND OF THE INVENTION

This invention relates to a ring gear used in an epicyclic gear train of a gas turbine engine.

Gas turbine engines typically employ an epicyclic gear train connected to the turbine section of the engine, which is used to drive the turbo fan. In a typical epicyclic gear train, a sun gear receives rotational input from a turbine shaft through a compressor shaft. A carrier supports intermediate gears that surround and mesh with the sun gear. A ring gear surrounds and meshes with the intermediate gears. In arrangements in which the carrier is fixed against rotation, the intermediate gears are referred to as “star” gears and the ring gear is coupled to an output shaft that supports the turbo fan.

Typically, the ring gear is connected to the turbo fan shaft using a spline ring. The spline ring is secured to a flange of the turbo fan shaft using circumferentially arranged bolts. The spline ring includes splines opposite the flange that supports a splined outer circumferential surface of the ring gear. The ring gear typically includes first and second portions that provide teeth facing in opposite directions, which mesh with complimentary oppositely facing teeth of the star gears.

An epicyclic gear train must share the load between the gears within the system. As a result, the splined connection between the ring gear and spline ring is subject to wear under high loads and deflection. Since the spline connection requires radial clearance, it is difficult to get a repeatable balance of the turbo fan assembly. Balance can also deteriorate over time with spline wear.

SUMMARY

In one exemplary embodiment, a gas turbine engine includes a bypass ratio greater than about ten (10). A fan is supported on a fan shaft and has a plurality of fan blades. There is a gutter with an annular channel. A gear system is connected to the fan shaft. There is a plurality of planetary gears and a ring gear with an aperture that is axially aligned with the annular channel. The ring gear includes a first portion with a first set of opposed angled teeth separated by a trough from a second portion with a second set of opposed angled teeth. A torque frame at least partially supports the gear system. A low pressure turbine has an inlet, an outlet,

and a low pressure turbine pressure ratio greater than 5:1 and a low fan pressure ratio of less than 1.45 across the fan blade alone.

In a further embodiment of any of the above, a support relative to a fixed housing facilitates segregation of vibrations.

In a further embodiment of any of the above, there is an input to the gear system that facilitates segregation of vibrations.

In a further embodiment of any of the above, the gear system includes a sun gear that has a sun gear splined connection.

In a further embodiment of any of the above, the input includes an input splined connection complementary to the sun gear splined connection. The gear system includes a gear reduction ratio of greater than 2.5.

In a further embodiment of any of the above, the fan shaft is supported by at least one tapered roller bearing.

In a further embodiment of any of the above, a fan tip speed less than 1150 ft/second.

In a further embodiment of any of the above, a fan tip speed is less than 1150 ft/second. A fan is on a fan shaft and has a plurality of fan blades and a low fan pressure ratio of less than 1.45 across the fan blades alone.

In a further embodiment of any of the above, the first portion and the second portion of the ring gear each include a radially extending flange that extends radially outward away from a corresponding set of opposed angled teeth. The fan shaft includes a radially extending flange connected to the radially extending flange on the first portion of the ring gear and the radially extending flange on the second portion of the ring gear.

In a further embodiment of any of the above, a seal is attached to at least one of the radially extending flanges on the ring gear or the radially extending flange on the fan shaft.

In a further embodiment of any of the above, the seal includes an oil return passage.

In a further embodiment of any of the above, the oil return passage includes a slot.

In a further embodiment of any of the above, the slot is located in the seal.

In a further embodiment of any of the above, the slot is located in the radially extending flange on the fan shaft.

In a further embodiment of any of the above, the gutter is for collecting oil expelled from the gear system.

In a further embodiment of any of the above, a fan tip speed less than 1150 ft/second.

In a further embodiment of any of the above, the gutter is formed from a soft material.

In a further embodiment of any of the above, the soft material is aluminum. In another exemplary embodiment, a gas turbine engine includes a fan supported on a fan shaft by at least one roller bearing and has a plurality of fan blades.

A bypass ratio is greater than about ten (10). There is a gutter with an annular channel. A planetary gear system is connected to the fan shaft. The gear system has a gear reduction ratio of greater than 2.5 and a plurality of intermediate gears. A ring gear with an aperture is axially aligned with the annular channel. The ring gear includes a first portion separated from a second portion. The first portion and the second portion each include a radially extending flange that extends radially outward away from a corresponding set of opposed angled teeth. A torque frame is at least partially supporting the gear system with respect to the housing which facilitates the segregation of vibrations and other transients.

In a further embodiment of any of the above, there is an input to the gear system which facilitates segregation of vibrations.

In a further embodiment of any of the above, the gear system includes a sun gear that has a sun gear splined connection. The input includes an input splined connection complementary to the sun gear splined connection.

In a further embodiment of any of the above, the roller bearing is a tapered roller bearing.

In a further embodiment of any of the above, the gutter is for collecting oil expelled from the gear system.

In a further embodiment of any of the above, the gutter is formed from a soft material.

In a further embodiment of any of the above, the soft material is aluminum.

In a further embodiment of any of the above, a low pressure turbine has an inlet, an outlet, and a low pressure turbine pressure ratio greater than 5:1. A low fan pressure ratio of less than 1.45 which is measured across the fan blades alone.

In a further embodiment of any of the above, the fan shaft includes a radially extending flange which is connected to the radially extending flange on the first portion of the ring gear and the radially extending flange on the second portion of the ring gear.

In a further embodiment of any of the above, a seal is attached to at least one of the radially extending flanges on the ring gear or the radially extending flange on the fan shaft.

In a further embodiment of any of the above, the seal includes an oil return passage.

BRIEF DESCRIPTION OF THE DRAWINGS

FIG. 1 is a partial cross-sectional view of a front portion of a gas turbine engine illustrating a turbo fan, epicyclic gear train and a compressor section.

FIG. 2 is an enlarged cross-sectional view of the epicyclic gear train shown in FIG. 1.

FIG. 3 is an enlarged cross-sectional view of an example ring gear similar to the arrangement shown in FIG. 2.

FIG. 4 is a view of the ring gear shown in FIG. 3 viewed in a direction that faces the teeth of the ring gear in FIG. 3.

FIG. 5 shows another embodiment.

FIG. 6 shows yet another embodiment.

DETAILED DESCRIPTION

A portion of a gas turbine engine 10 is shown schematically in FIG. 1. The turbine engine 10 includes a fixed housing 12 that is constructed from numerous pieces secured to one another. A compressor section 14 having compressor hubs 16 with blades are driven by a turbine shaft 25 about an axis A. A turbo fan 18 is supported on a turbo fan shaft 20 that is driven by a compressor shaft 24, which supports the compressor hubs 16, through an epicyclic gear train 22. The engine 10 is a high-bypass geared architecture aircraft engine. In one disclosed, non-limiting embodiment, the engine 10 bypass ratio is greater than about six (6) to ten (10), the gear train 22 is an epicyclic gear train such as a planetary gear system or other gear system with a gear reduction ratio of greater than about 2.3 and the low pressure turbine 18 has a pressure ratio that is greater than about 5. The engine 10 in one non-limiting embodiment is a high-bypass geared architecture aircraft engine. In one disclosed embodiment, the engine 10 bypass ratio is greater than ten (10:1), the turbofan diameter is significantly larger than that of the low pressure compressor 16, and the low pressure

turbine 27a (shown schematically) has a pressure ratio that is greater than 5:1. As understood, the low pressure turbine 27a is downstream of at least one upstream, or high pressure turbine. The gear train 22 may be an epicyclic gear train such as a planetary gear system or other gear system with a gear reduction ratio of greater than 2.5:1. It should be understood, however, that the above parameters are only exemplary of one embodiment of a geared architecture engine and that the present invention is applicable to other gas turbine engines including direct drive turbofans.

A significant amount of thrust is provided by the bypass flow B due to the high bypass ratio. The fan 18 of the engine 10 is designed for a particular flight condition—typically cruise at about 0.8 M and about 35,000 feet. The flight condition of 0.8 M and 35,000 ft, with the engine at its best fuel consumption—also known as “bucket cruise TSFC”—is the industry standard parameter of lbf of fuel being burned divided by lbf of thrust the engine produces at that minimum point. “Low fan pressure ratio” is the pressure ratio across the fan blade alone, without the FEGV system 36. The low fan pressure ratio as disclosed herein according to one non-limiting embodiment is less than 1.45. “Low corrected fan tip speed” is the actual fan tip speed in ft/sec divided by an industry standard temperature correction of $[(T_{\text{ambient}} \text{ deg R})/518.7]^{0.5}$. The “Low corrected fan tip speed” as disclosed herein according to one non-limiting embodiment is less than 1150 ft/second.

The gear train 22 generally includes a fan drive gear system (FDGS) 100 driven by the compressor shaft 24 through an input coupling 102. The input coupling 102 both transfers torque from the compressor shaft 24 to the gear train 22 and facilitates the segregation of vibrations and other transients therebetween.

The input coupling 102 may include an interface spline 102 joined, by a gear spline 106, to the sun gear 30. The sun gear 30 is in meshed engagement with multiple star gears 32. Each star gear 32 is also in meshed engagement with rotating ring gear 38 that is mechanically connected to the fan shaft 20. Since the star gears 32 mesh with both the rotating ring gear 38 as well as the rotating sun gear 30, the star gears 32 rotate about their own axes to drive the ring gear 38. The rotation of the ring gear 38 is conveyed to the fan 20 through the fan shaft 20 to thereby drive the fan 18 at a lower speed than the turbine shaft 25.

In the example arrangement shown, the epicyclic gear train 22 is a star gear train. Referring to FIG. 2, the epicyclic gear train 22 includes the sun gear 30 that is connected to the compressor shaft 24, which provides rotational input, by the splined connection. A carrier 26 is fixed to the housing 12 by a torque frame 28 using fingers (not shown) known in the art. The carrier 26 supports star gears 32 using journal bearings 34 that are coupled to the sun gear 30 by meshed interfaces between the teeth of sun and star gears 30, 32. Multiple star gears 32 are arranged circumferentially about the sun gear 30. Retainers 36 retain the journal bearings 34 to the carrier 26. A ring gear 38 surrounds the carrier 26 and is coupled to the star gears 32 by meshed interfaces. The ring gear 38, which provides rotational output, is secured to the turbo fan shaft 20 by circumferentially arranged fastening elements, which are described in more detail below.

The torque frame 28 supports the carrier 26 with respect to the housing 12 such as a front center body which facilitates the segregation of vibrations and other transients therebetween. It should be understood that various gas turbine engine case structures may alternatively or additionally be provided.

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The fixed housing **12** may further include a number 1 and 1.5 bearing support frame **108** which is commonly referred to as a “K-frame” which supports the number 1 and number 1.5 bearing systems **110A**, **110B** to support the fan shaft **20** (FIG. 1). The number 1 and number 1.5 bearing systems **110A**, **110B** may include tapered roller bearings which provide a line contact.

Referring to FIGS. 3 and 4, the ring gear **38** is a two-piece construction having first and second portions **40**, **42**. The first and second portions **40**, **42** abut one another at a radial interface **45**. A trough **41** separates oppositely angled teeth **43** (best shown in FIG. 4) on each of the first and second portions **40**, **42**. The arrangement of teeth **43** forces the first and second portions **40**, **42** toward one another at the radial interface **45**. The back side of the first and second portions **40**, **42** includes a generally S-shaped outer circumferential surface **47** that, coupled with a change in thickness, provides structural rigidity and resistance to overturning moments. The first and second portions **40**, **42** have a first thickness **T1** that is less than a second thickness **T2** arranged axially inwardly from the first thickness **T1**. The first and second portions **40**, **42** include facing recesses **44** that form an internal annular cavity **46**.

The first and second portions **40**, **42** include flanges **51** that extend radially outward away from the teeth **43**. The turbo fan shaft **20** includes a radially outwardly extending flange **70** that is secured to the flanges **51** by circumferentially arranged bolts **52** and nuts **54**, which axially constrain and affix the turbo fan shaft **20** and ring gear **38** relative to one another. Thus, the spline ring is eliminated, which also reduces heat generated from windage and churning that resulted from the sharp edges and surface area of the splines. The turbo fan shaft **20** and ring gear **38** can be rotationally balanced with one another since radial movement resulting from the use of splines is eliminated. An oil baffle **68** is also secured to the flanges **51**, **70** and balanced with the assembly.

Seals **56** having knife edges **58** are secured to the flanges **51**, **70**. The first and second portions **40**, **42** have grooves **48** at the radial interface **45** that form a hole **50**, which expels oil through the ring gear **38** to a gutter **60** that is secured to the carrier **26** with fasteners **61** (FIG. 2). The direct radial flow path provided by the grooves **48** reduces windage and churning by avoiding the axial flow path change that existed with splines. That is, the oil had to flow radially and then axially to exit through the spline interface. The gutter **60** is constructed from a soft material such as aluminum so that the knife edges **58**, which are constructed from steel, can cut into the aluminum if they interfere. Referring to FIG. 3, the seals **56** also include oil return passages **62** provided by first and second slots **64** in the seals **56**, which permit oil on either side of the ring gear **38** to drain into the gutter **60**. In the example shown in FIG. 2, the first and second slots **64**, **66** are instead provided in the flange **70** and oil baffle **68**, respectively.

FIG. 5 shows an embodiment **200**, wherein there is a fan drive turbine **208** driving a shaft **206** to in turn drive a fan rotor **202**. A gear reduction **204** may be positioned between the fan drive turbine **208** and the fan rotor **202**. This gear reduction **204** may be structured and operate like the gear reduction disclosed above. A compressor rotor **210** is driven by an intermediate pressure turbine **212**, and a second stage compressor rotor **214** is driven by a turbine rotor **216**. A combustion section **218** is positioned intermediate the compressor rotor **214** and the turbine section **216**.

FIG. 6 shows yet another embodiment **300** wherein a fan rotor **302** and a first stage compressor **304** rotate at a

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common speed. The gear reduction **306** (which may be structured as disclosed above) is intermediate the compressor rotor **304** and a shaft **308** which is driven by a low pressure turbine section.

Although a preferred embodiment of this invention has been disclosed, a worker of ordinary skill in this art would recognize that certain modifications would come within the scope of this invention. For that reason, the following claims should be studied to determine the true scope and content of this invention.

What is claimed is:

1. A gas turbine engine comprising:

a bypass ratio greater than about ten (10);

a fan supported on a fan shaft and having a plurality of fan blades;

a gear system connected to the fan shaft, a plurality of planetary gears, a first set of opposed angled ring gear teeth separated from a second set of opposed angled ring gear teeth, wherein a lubricant flow path is located axially between the first set of opposed angled ring gear teeth and the second set of opposed angled ring gear teeth;

a gear system support relative to a fixed housing facilitating segregation of vibrations;

an annular channel axially aligned with the lubricant flow path; and

a low pressure turbine with an inlet, an outlet, and a low pressure turbine pressure ratio greater than 5:1.

2. The gas turbine engine of claim 1, further comprising a torque frame at least partially supporting the gear system, wherein the torque frame facilitates segregation of vibrations.

3. The gas turbine engine of claim 2, wherein the gear system includes a sun gear having a sun gear splined connection.

4. The gas turbine engine of claim 3, wherein said gear system includes a gear reduction ratio of greater than 2.5.

5. The gas turbine engine of claim 4, wherein the annular channel is at least partially defined by a gutter.

6. The gas turbine engine of claim 5, wherein the fan shaft is supported by at least one tapered roller bearing.

7. The gas turbine engine of claim 5, further comprising a fan tip speed less than 1150 ft/second.

8. The gas turbine engine of claim 7, further comprising a low fan pressure ratio of less than 1.45 across the fan blades alone.

9. The gas turbine engine of claim 5, wherein said gutter is for collecting oil expelled from the gear system.

10. The gas turbine engine of claim 9, further comprising a fan tip speed less than 1150 ft/second.

11. The gas turbine engine of claim 10, further comprising a low fan pressure ratio of less than 1.45 across the fan blades alone.

12. The gas turbine engine of claim 11, wherein the gutter is formed from a soft material.

13. The gas turbine engine of claim 12, wherein the soft material is aluminum.

14. The gas turbine engine of claim 8, wherein the first set of opposed angled ring gear teeth and the second set of opposed angled ring gear teeth each include a radially extending flange that extends radially outward away from a corresponding set of opposed angled ring gear teeth, and the fan shaft includes a radially extending flange connected to the radially extending flange on the first set of opposed angled ring gear teeth and the radially extending flange on the second set of opposed angled ring gear teeth.

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15. The gas turbine engine of claim 14, further comprising a seal attached to at least one of the radially extending flanges on the first and second set of opposed angled ring gear teeth or the radially extending flange on the fan shaft.

16. The gas turbine engine of claim 15, wherein the seal includes an oil return passage.

17. The gas turbine engine of claim 16, wherein the oil return passage includes a slot.

18. The gas turbine engine of claim 17, wherein the slot is located in the seal.

19. The gas turbine engine of claim 17, wherein the slot is located in the radially extending flange on the fan shaft.

20. The gas turbine engine of claim 3, further comprising an input to the gear system, wherein the input facilitates segregation of vibrations and includes an input connection complementary to the sun gear connection.

21. A gas turbine engine comprising:

a fan supported on a fan shaft by at least one roller bearing and having a plurality of fan blades;

a bypass ratio greater than about ten (10);

an epicyclic gear system connected to the fan shaft, the epicyclic gear system having a gear reduction ratio of greater than 2.5, a plurality of intermediate gears, and a ring gear with an aperture that is, wherein the ring gear includes a first portion separated from a second portion, the first portion and the second portion each include a radially extending flange that extends radially outward away from a corresponding set of opposed angled teeth;

a torque frame as least partially supporting the gear system with respect to a housing to facilitate segregation of vibrations; and

an annular channel axially aligned with the aperture and located radially outward from the aperture.

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22. The gas turbine engine of claim 21, wherein the gear system includes a sun gear having a sun gear connection.

23. The gas turbine engine of claim 22, wherein the annular channel includes a gutter for collecting oil expelled from the gear system.

24. The gas turbine engine of claim 23, wherein the gutter is formed from aluminum.

25. The gas turbine of claim 24, further comprising a low pressure turbine with an inlet, an outlet, and a low pressure turbine pressure ratio greater than 5:1, and a low fan pressure ratio of less than 1.45 and the low fan pressure ratio is measured across the fan blades alone.

26. The gas turbine of claim 24, further comprising a low pressure turbine with an inlet, an outlet, and a low pressure turbine pressure ratio greater than 5:1, a low fan pressure ratio of less than 1.45 and the low fan pressure ratio is measured across the fan blades alone, and a gear system support relative to a fixed housing facilitating segregation of vibrations.

27. The gas turbine engine of claim 26, wherein the fan shaft includes a radially extending flange connected to the radially extending flange on the first portion of the ring gear and the radially extending flange on the second portion of the ring gear.

28. The gas turbine engine of claim 27, further comprising a seal attached to at least one of the radially extending flanges on the ring gear or the radially extending flange on the fan shaft, wherein the seal includes an oil return passage.

29. The gas turbine engine of claim 28, further comprising a fan tip speed less than 1150 ft/second.

30. The gas turbine engine of claim 29, further comprising an input to the gear system wherein the input facilitates segregation of vibrations and includes an input connection complementary to the sun gear connection.

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