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(54) **FILLET OPTIMIZATION FOR TURBINE AIRFOIL**

(58) **Field of Classification Search**
CPC ... F01D 5/18; F01D 5/186; F01D 5/20; F01D 5/288; F01D 5/187

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See application file for complete search history.

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This patent is subject to a terminal disclaimer.

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Related U.S. Application Data

(57) **ABSTRACT**

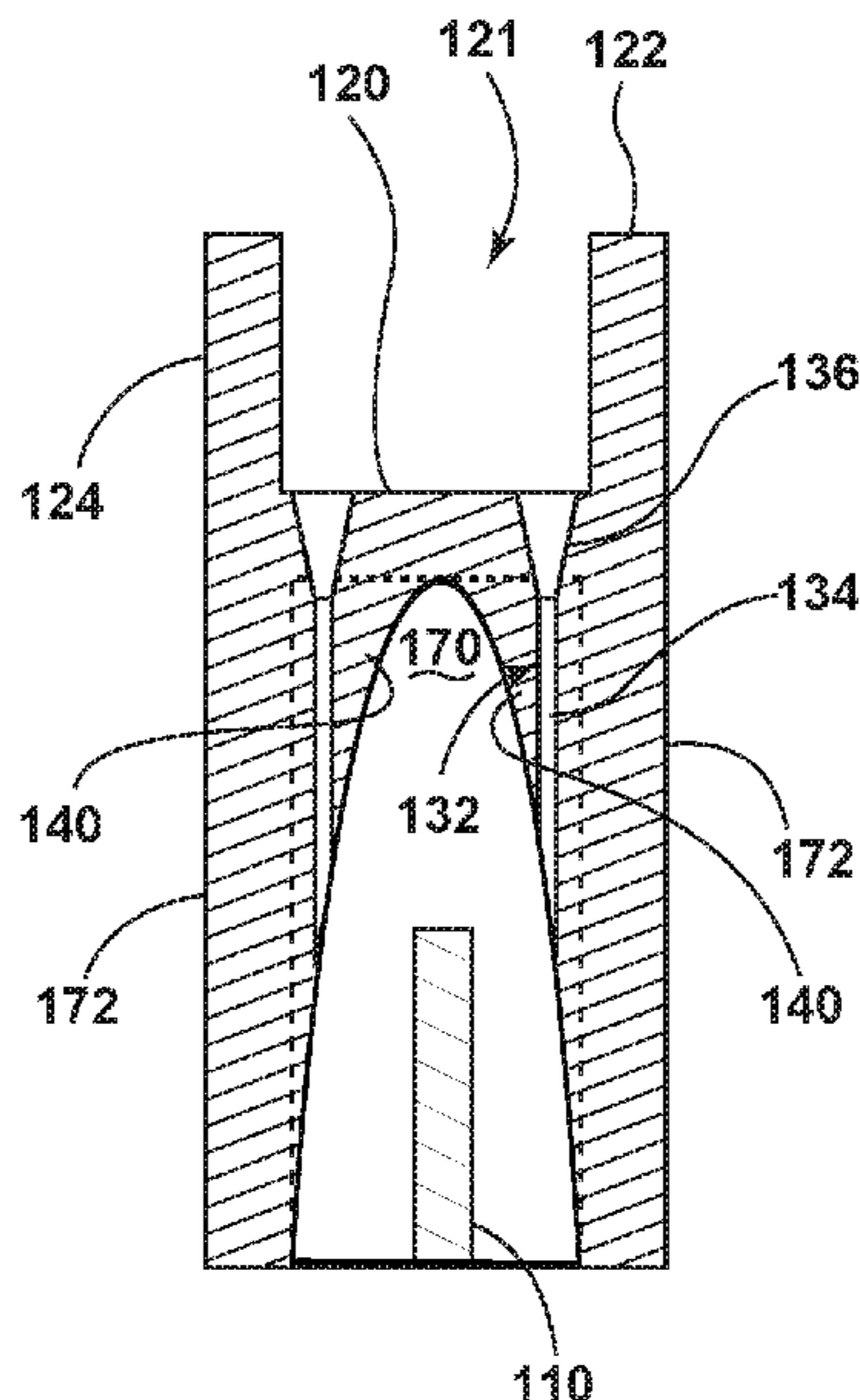
(62) Division of application No. 14/960,991, filed on Dec. 7, 2015, now Pat. No. 10,227,876.

A blade for a gas turbine engine comprises an airfoil having a pressure side and a suction side, with a root and a tip wall. The pressure side and suction side extend beyond the tip wall to define a tip channel, defining a plurality of internal and external corners. The corners comprise fillets to define a thickness being greater than the thickness for the pressure, suction, or tip walls. A film hole can extend through the fillet, such that the length of the film hole at the fillet can be increased to define an increased length-to-diameter ratio for the film hole to improve film cooling through the film hole.

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(Continued)

27 Claims, 10 Drawing Sheets



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 (2013.01); *Y02T 50/60* (2013.01)

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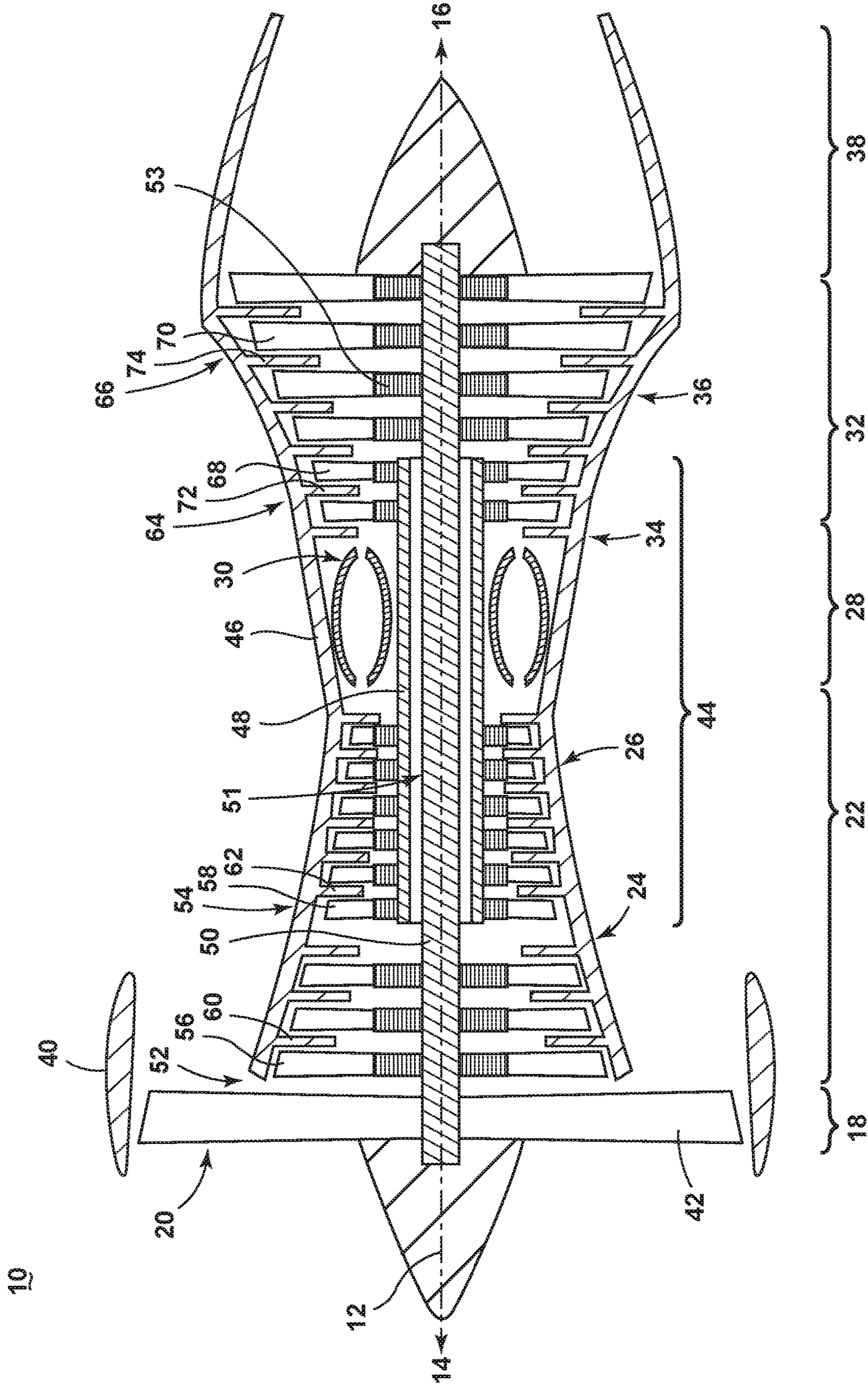


FIG. 1

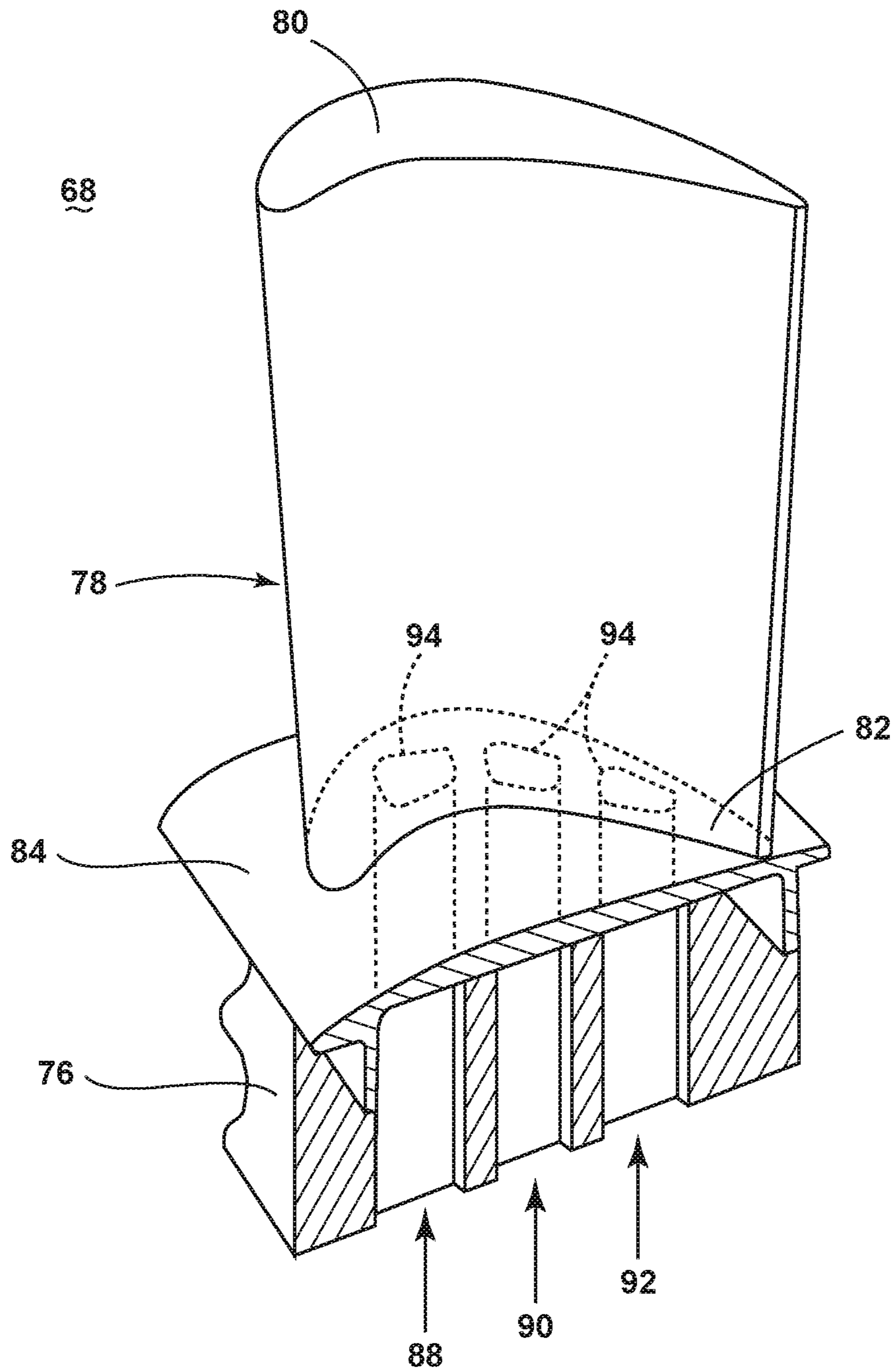


FIG. 2

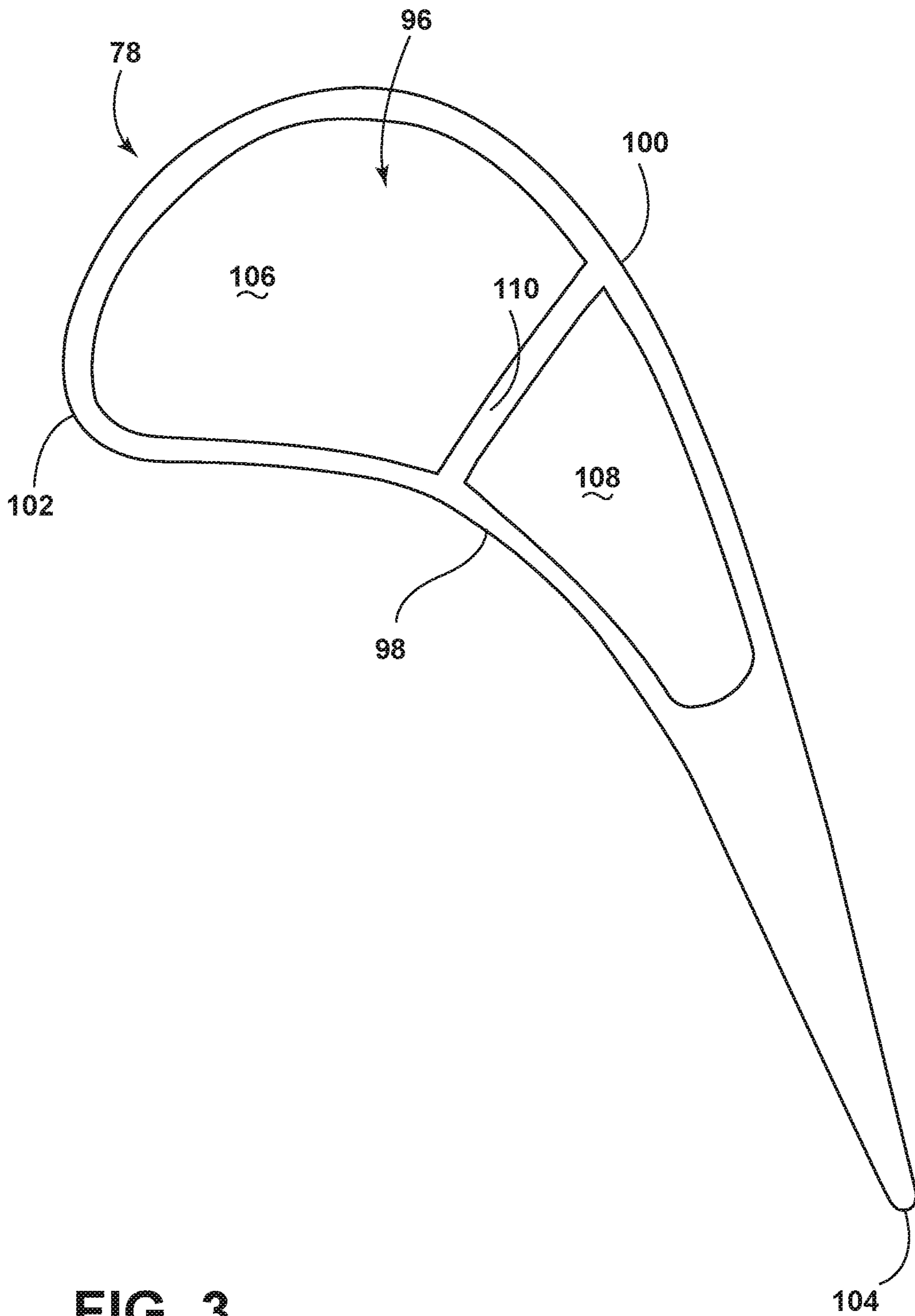


FIG. 3

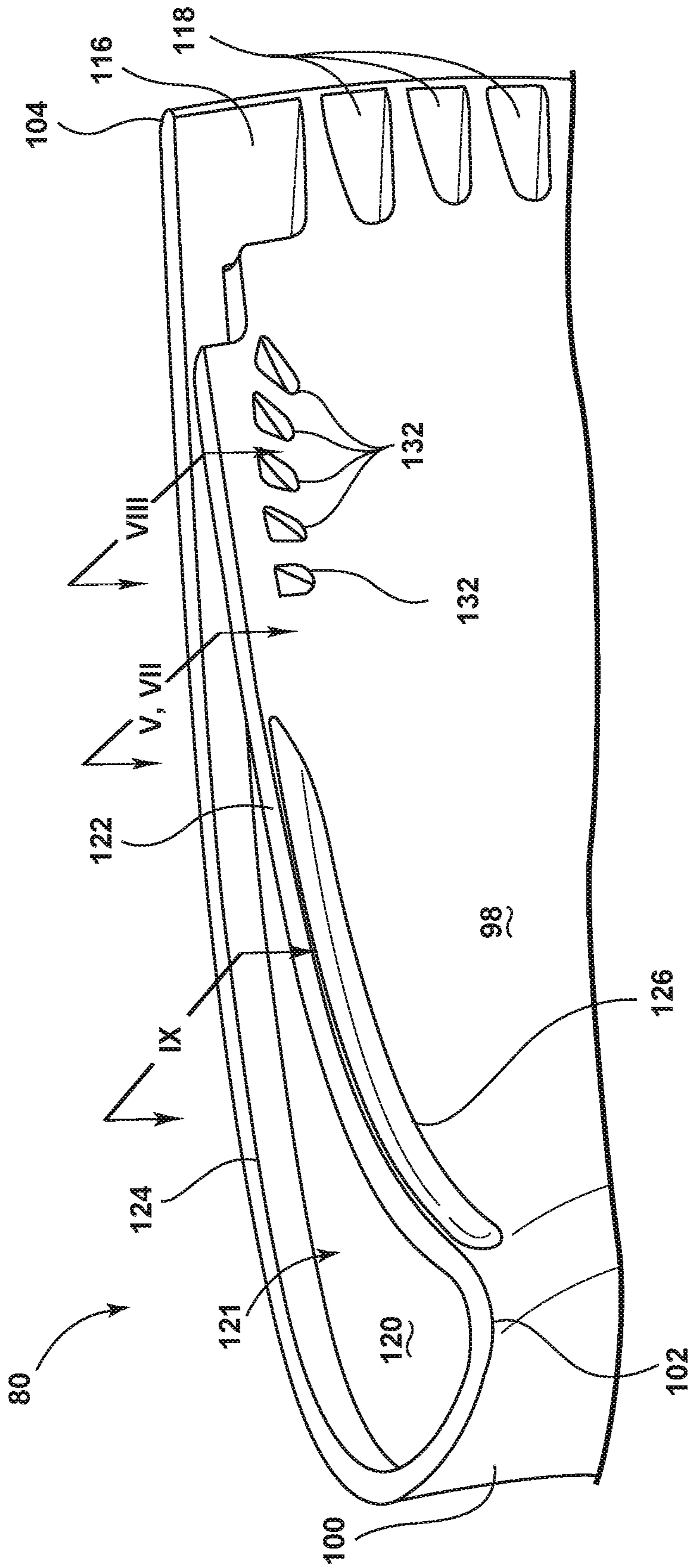


FIG. 4

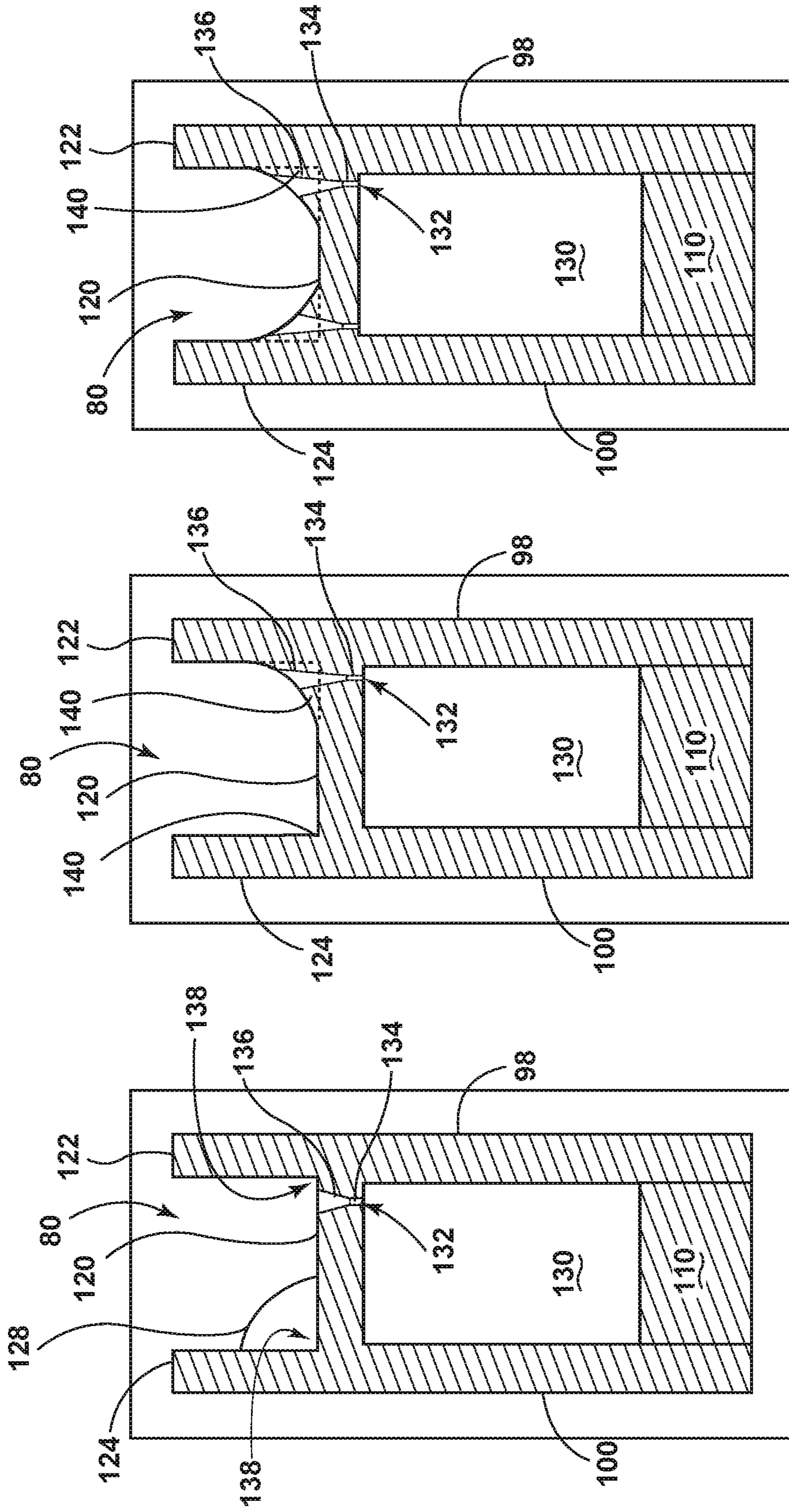


FIG. 5A
(Prior Art)

FIG. 5B

FIG. 5C

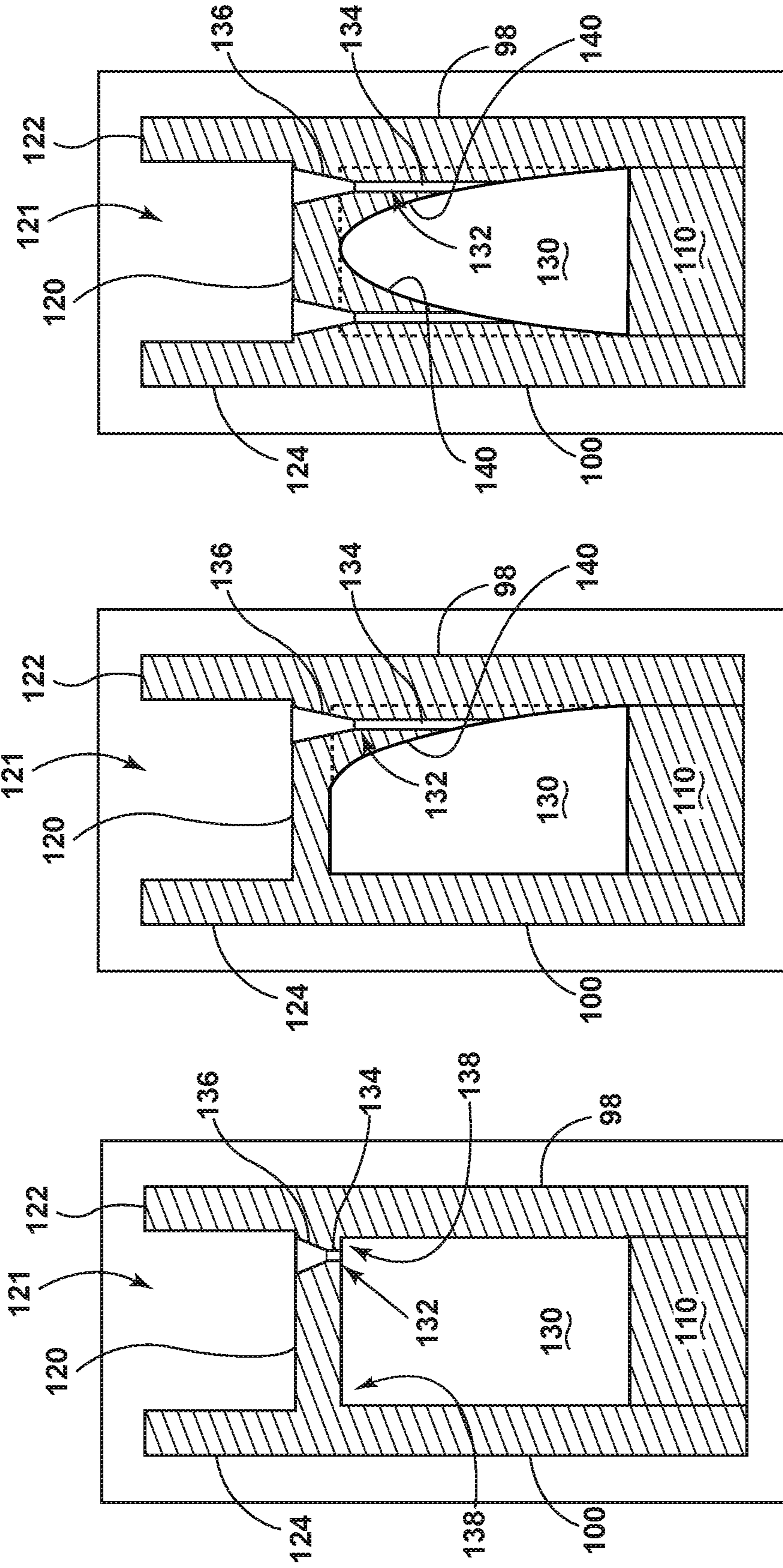


FIG. 7A
(Prior Art)

FIG. 7B

FIG. 7C

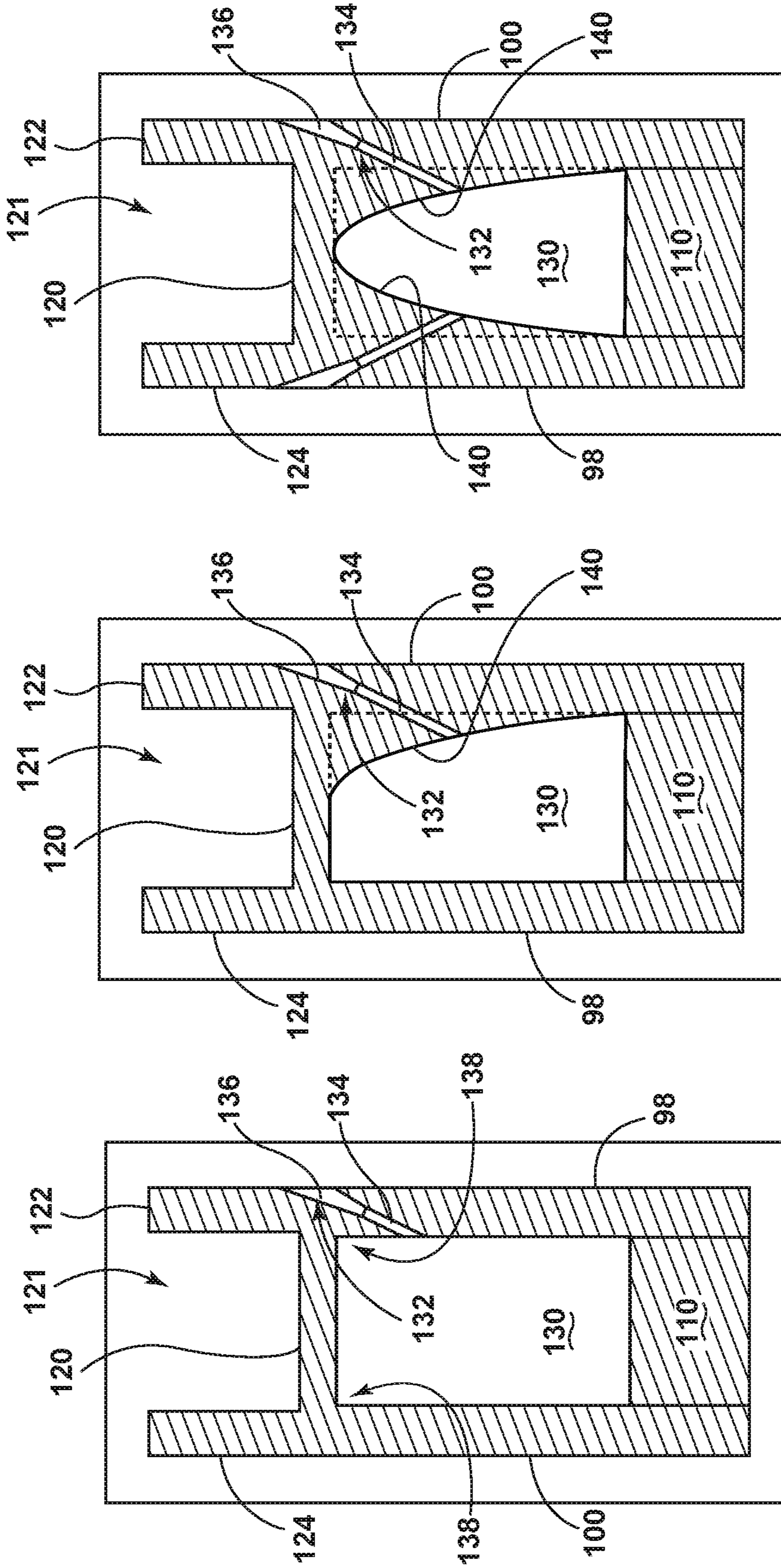


FIG. 8A
(Prior Art)

FIG. 8B

FIG. 8C

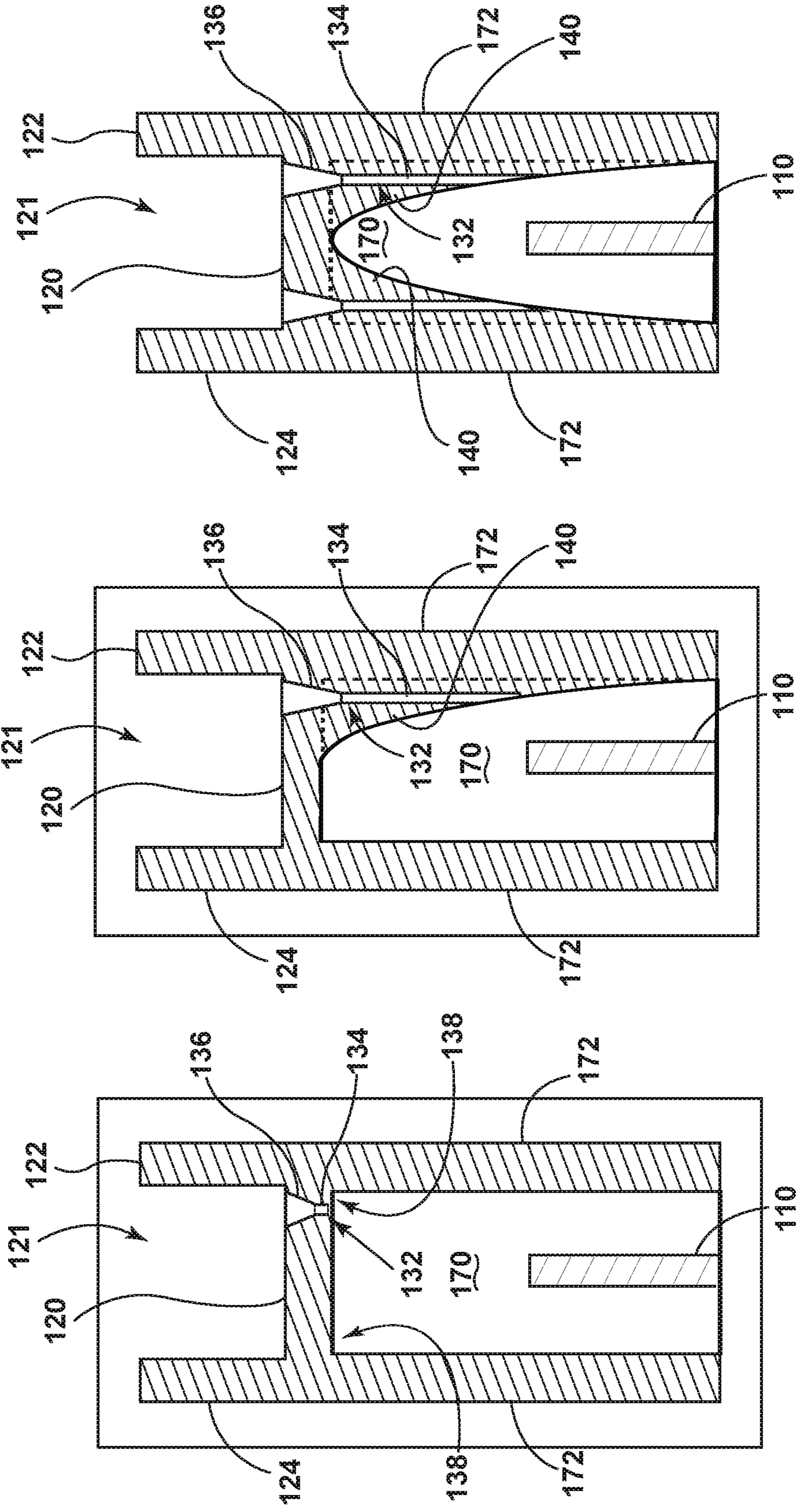


FIG. 9A
(Prior Art)

FIG. 9B

FIG. 9C

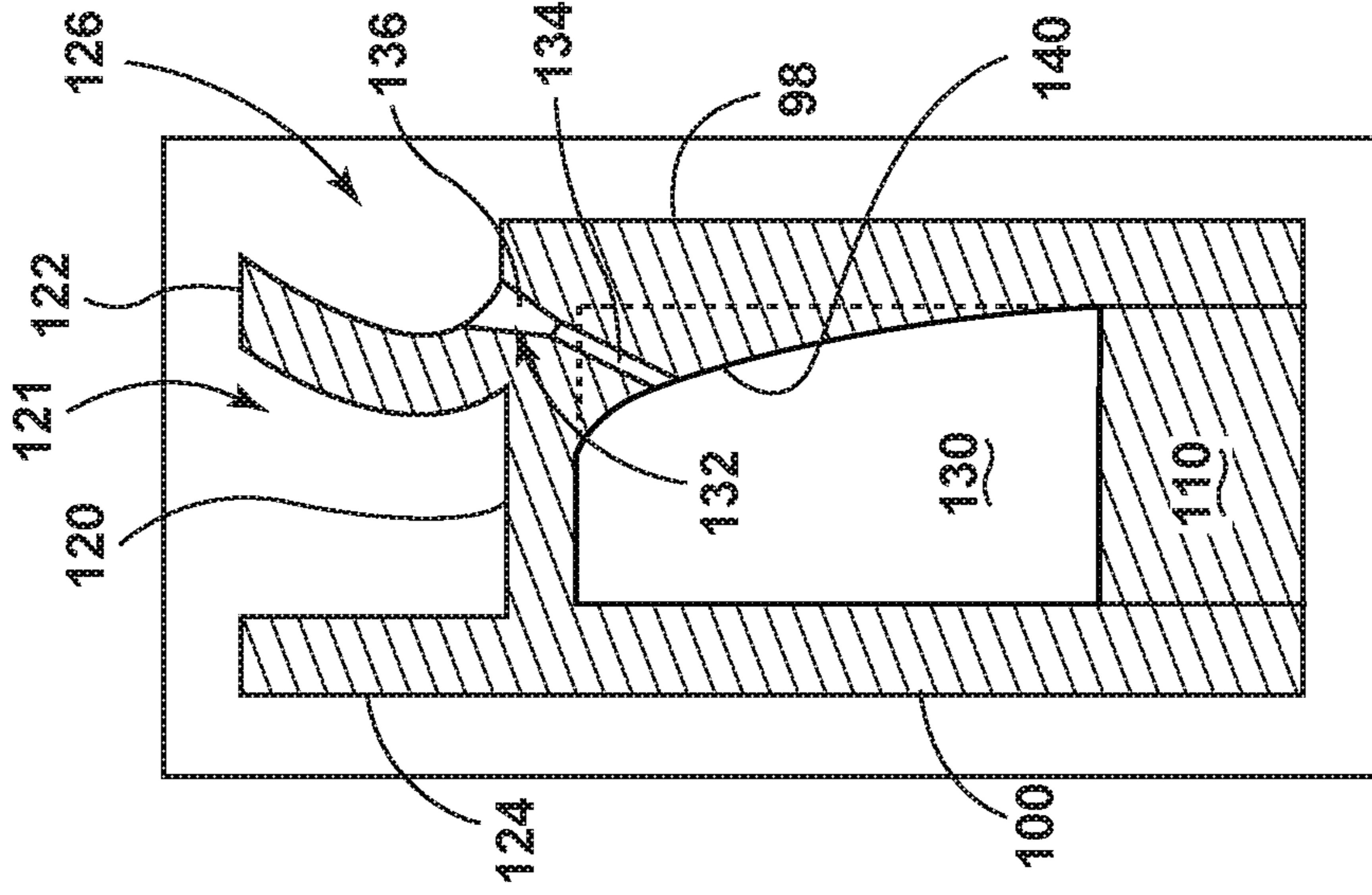


FIG. 10A

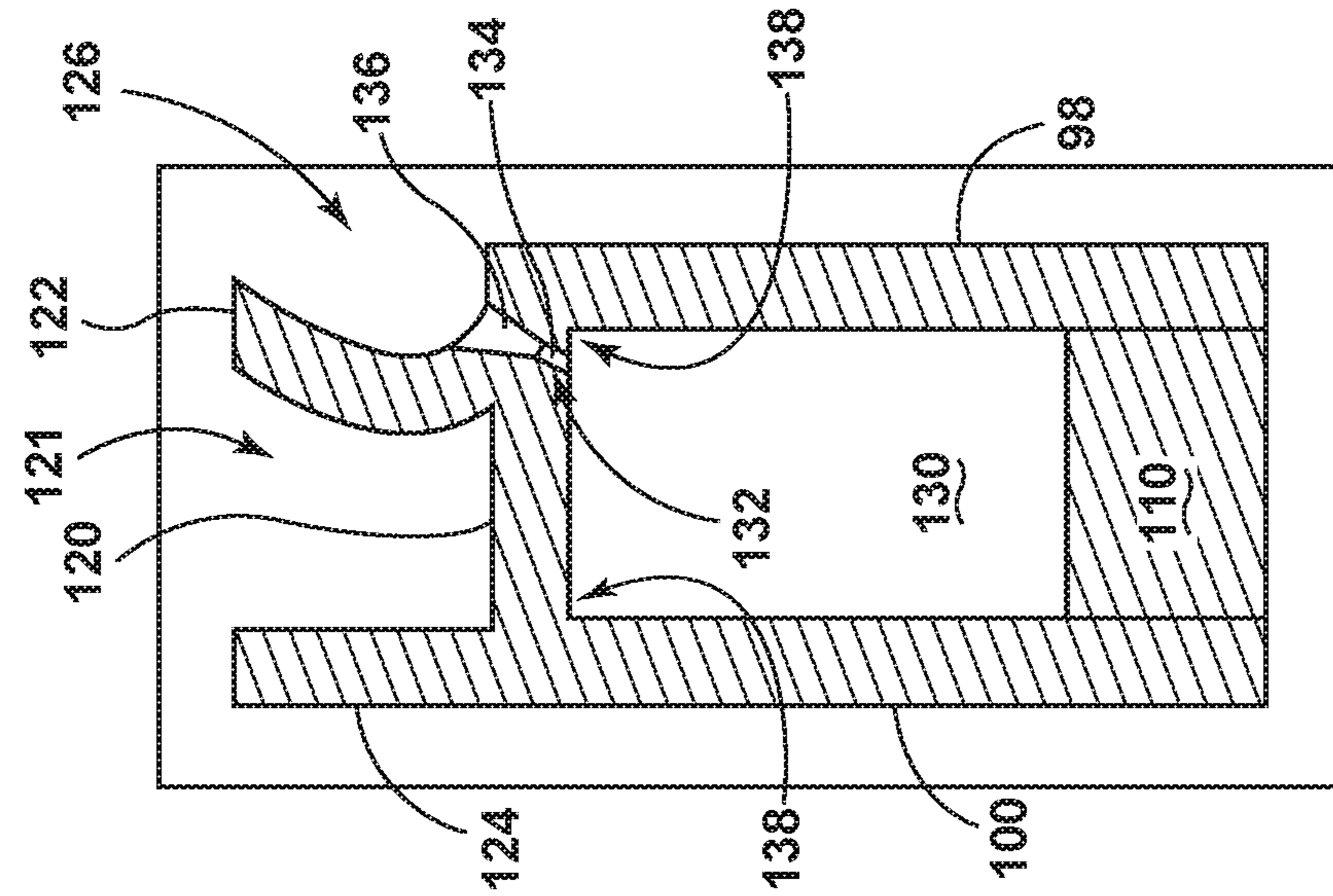


FIG. 10B

1

FILLET OPTIMIZATION FOR TURBINE AIRFOIL

CROSS-REFERENCE TO RELATED APPLICATIONS

This application is a division of U.S. application Ser. No. 14/960,991, filed on Dec. 7, 2015, titled "FILLET OPTIMIZATION FOR TURBINE AIRFOIL", which is herein incorporated by reference.

BACKGROUND OF THE INVENTION

Turbine engines, and particularly gas or combustion turbine engines, are rotary engines that extract energy from a flow of combusted gases passing through the engine in a series of compressor stages, which include pairs of rotating blades and stationary vanes, through a combustor, and then onto a multitude of turbine blades. Gas turbine engines have been used for land and nautical locomotion and power generation, but are most commonly used for aeronautical applications such as for airplanes, including helicopters. In airplanes, gas turbine engines are used for propulsion of the aircraft.

Gas turbine engines for aircraft are designed to operate at high temperatures to maximize engine efficiency, so cooling of certain engine components, such as the high pressure turbine and the low pressure turbine, can be beneficial. Typically, cooling is accomplished by ducting cooler air from the high and/or low pressure compressors to the engine components that require cooling. Temperatures in the high pressure turbine are around 1000° C. to 2000° C. and the cooling air from the compressor is around 500° C. to 700° C. While the compressor air is a high temperature, it is cooler relative to the turbine air, and can be used to cool the turbine.

Contemporary turbine blades, as well as vanes or nozzles, generally include one or more interior cooling circuits for routing the cooling air through the blade to cool different portions of the blade, and can include dedicated cooling circuits for cooling different portions of the blade, such as the leading edge, trailing edge and tip of the blade.

BRIEF DESCRIPTION OF THE INVENTION

In one aspect, a blade for a gas turbine engine comprising an airfoil having an outer wall defining a pressure side and a suction side, the outer wall extending chord-wise from a leading edge to a trailing edge and span-wise from a root toward a tip. The blade further comprises a tip channel spanning the pressure side and the suction side of the outer wall and intersecting the outer wall to form at least one corner, with the outer wall having a first thickness at the corner and the tip channel having a second thickness at the corner. The blade further has a cooling passage having a portion located along the tip channel and at least partially defined by the tip channel and the outer wall such that the corner defines a corner of the cooling passage. Further still, the blade comprises a fillet located at the corner having an effective radius of at least 1.5 times larger than the greater of the first and second thicknesses and at least one film hole extending through the fillet to fluidly couple the cooling passage to an exterior of the airfoil.

In another aspect, a method of forming a film holes in a blade of a gas turbine engine comprising forming the film hole through a fillet of a corner of a cooling passage formed by the intersection of a tip channel and an outer wall, with

2

the fillet having an effective radius of at least 1.5 times greater than the greater of the thicknesses for the tip channel and the outer wall.

In another aspect, a blade for a gas turbine engine comprising an airfoil having an internal cooling passage at least partially formed by intersecting a tip channel and an outer wall defining a corner having a fillet with an effective radius of at least 1.5 times the thickness of the thickest of the intersecting tip channel and outer wall and at least one film hole extending through the fillet and at least one of the tip channel and outer wall.

BRIEF DESCRIPTION OF THE DRAWINGS

In the drawings:

FIG. 1 is a schematic cross-sectional diagram of a gas turbine engine for an aircraft.

FIG. 2 is a perspective view of an engine component in the form of a turbine blade of the engine of FIG. 1 with cooling air inlet passages.

FIG. 3 is a cross-sectional view of the airfoil of FIG. 2.

FIG. 4 is a perspective view of a tip of the airfoil of FIG. 2 illustrating a tip channel and a tip shelf.

FIGS. 5A-5C are cross-sectional views of section V of FIG. 4 illustrating an increased film hole length by placing a fillet in the tip channel.

FIG. 6 is a schematic view illustrating the increased film hole length with an external fillet at the tip of the airfoil.

FIGS. 7A-7C are cross-sectional views of section VII of FIG. 4 illustrating the increased film hole length by placing a fillet internal of the airfoil adjacent to the tip channel.

FIGS. 8A-8C are cross-sectional views of section VIII of FIG. 4 illustrating the increased film hole length with an internal fillet having the film hole extending to a sidewall.

FIGS. 9A-9C are cross-sectional views of section VIII of FIG. 4 illustrating the increased film hole length with an internal fillet within an internal serpentine circuit.

FIGS. 10A-10B are cross-sectional views of section IX of FIG. 4 illustrating the increased film hole length with an internal fillet having the film hole extending to a tip shelf.

DESCRIPTION OF EMBODIMENTS OF THE INVENTION

The described embodiments of the present invention are directed to apparatuses, methods, and other devices related to routing airflow in a turbine engine. For purposes of illustration, the present invention will be described with respect to an aircraft gas turbine engine. It will be understood, however, that the invention is not so limited and can have general applicability in non-aircraft applications, such as other mobile applications and non-mobile industrial, commercial, and residential applications.

It should be further understood that for purposes of illustration, the present invention will be described with respect to an airfoil for a turbine blade of the turbine engine. It will be understood, however, that the invention is not limited to the turbine blade, and can comprise any airfoil structure, such as a compressor blade, a turbine or compressor vane, a fan blade, or a strut in non-limiting examples. Furthermore, the filleted optimization can have uses in additional engine components utilizing film holes or surface film cooling, such as a band, combustor assembly, or platform in non-limiting examples.

As used herein, the term "forward" or "upstream" refers to moving in a direction toward the engine inlet, or a component being relatively closer to the engine inlet as

compared to another component. The term “aft” or “downstream” used in conjunction with “forward” or “upstream” refers to a direction toward the rear or outlet of the engine relative to the engine centerline.

Additionally, as used herein, the terms “radial” or “radially” refer to a dimension extending between a center longitudinal axis of the engine and an outer engine circumference.

All directional references (e.g., radial, axial, proximal, distal, upper, lower, upward, downward, left, right, lateral, front, back, top, bottom, above, below, vertical, horizontal, clockwise, counterclockwise, upstream, downstream, aft, etc.) are only used for identification purposes to aid the reader’s understanding of the present invention, and do not create limitations, particularly as to the position, orientation, or use of the invention. Connection references (e.g., attached, coupled, connected, and joined) are to be construed broadly and can include intermediate members between a collection of elements and relative movement between elements unless otherwise indicated. As such, connection references do not necessarily infer that two elements are directly connected and in fixed relation to one another. The exemplary drawings are for purposes of illustration only and the dimensions, positions, order and relative sizes reflected in the drawings attached hereto can vary.

It should be further understood that as used herein, the term ‘fillet’ is used to describe material that “fills” in a corner formed by a junction of two intersecting walls. It should be further understood that the intersecting walls can be integral and need not comprise separate intersecting elements. Similarly, the fillet can be integral with the intersecting walls. In the case of the fillet being integral to the intersecting walls, there is no clear demarcation between the fillet and the corner. In such a case, the fillet can be identified by virtually extending the wall thickness until the walls intersect to form a virtual corner.

FIG. 1 is a schematic cross-sectional diagram of a gas turbine engine 10 for an aircraft. The engine 10 has a generally longitudinally extending axis or centerline 12 extending forward 14 to aft 16. The engine 10 includes, in downstream serial flow relationship, a fan section 18 including a fan 20, a compressor section 22 including a booster or low pressure (LP) compressor 24 and a high pressure (HP) compressor 26, a combustion section 28 including a combustor 30, a turbine section 32 including a HP turbine 34, and a LP turbine 36, and an exhaust section 38.

The fan section 18 includes a fan casing 40 surrounding the fan 20. The fan 20 includes a plurality of fan blades 42 disposed radially about the centerline 12. The HP compressor 26, the combustor 30, and the HP turbine 34 form a core 44 of the engine 10, which generates combustion gases. The core 44 is surrounded by core casing 46, which can be coupled with the fan casing 40.

A HP shaft or spool 48 disposed coaxially about the centerline 12 of the engine 10 drivingly connects the HP turbine 34 to the HP compressor 26. A LP shaft or spool 50, which is disposed coaxially about the centerline 12 of the engine 10 within the larger diameter annular HP spool 48, drivingly connects the LP turbine 36 to the LP compressor 24 and fan 20. The portions of the engine 10 mounted to and rotating with either or both of the spools 48, 50 are referred to individually or collectively as a rotor 51.

The LP compressor 24 and the HP compressor 26 respectively include a plurality of compressor stages 52, 54, in which a set of compressor blades 58 rotate relative to a corresponding set of static compressor vanes 60, 62 (also called a nozzle) to compress or pressurize the stream of fluid

passing through the stage. In a single compressor stage 52, 54, multiple compressor blades 56, 58 can be provided in a ring and can extend radially outwardly relative to the centerline 12, from a blade platform to a blade tip, while the corresponding static compressor vanes 60, 62 are positioned downstream of and adjacent to the rotating blades 56, 58. It is noted that the number of blades, vanes, and compressor stages shown in FIG. 1 were selected for illustrative purposes only, and that other numbers are possible. The blades 56, 58 for a stage of the compressor can be mounted to a disk 53, which is mounted to the corresponding one of the HP and LP spools 48, 50, with each stage having its own disk. The vanes 60, 62 are mounted to the core casing 46 in a circumferential arrangement about the rotor 51.

The HP turbine 34 and the LP turbine 36 respectively include a plurality of turbine stages 64, 66, in which a set of turbine blades 68, 70 are rotated relative to a corresponding set of static turbine vanes 72, 74 (also called a nozzle) to extract energy from the stream of fluid passing through the stage. In a single turbine stage 64, 66, multiple turbine blades 68, 70 can be provided in a ring and can extend radially outwardly relative to the centerline 12, from a blade platform to a blade tip, while the corresponding static turbine vanes 72, 74 are positioned upstream of and adjacent to the rotating blades 68, 70. It is noted that the number of blades, vanes, and turbine stages shown in FIG. 1 were selected for illustrative purposes only, and that other numbers are possible.

In operation, the rotating fan 20 supplies ambient air to the LP compressor 24, which then supplies pressurized ambient air to the HP compressor 26, which further pressurizes the ambient air. The pressurized air from the HP compressor 26 is mixed with fuel in the combustor 30 and ignited, thereby generating combustion gases. Some work is extracted from these gases by the HP turbine 34, which drives the HP compressor 26. The combustion gases are discharged into the LP turbine 36, which extracts additional work to drive the LP compressor 24, and the exhaust gas is ultimately discharged from the engine 10 via the exhaust section 38. The driving of the LP turbine 36 drives the LP spool 50 to rotate the fan 20 and the LP compressor 24.

Some of the ambient air supplied by the fan 20 can bypass the engine core 44 and be used for cooling of portions, especially hot portions, of the engine 10, and/or used to cool or power other aspects of the aircraft. In the context of a turbine engine, the hot portions of the engine are normally downstream of the combustor 30, especially the turbine section 32, with the HP turbine 34 being the hottest portion as it is directly downstream of the combustion section 28. Other sources of cooling fluid can be, but is not limited to, fluid discharged from the LP compressor 24 or the HP compressor 26.

FIG. 2 is a perspective view of an engine component in the form of one of the turbine blades 68 of the engine 10 from FIG. 1. The turbine blade 68 includes a dovetail 76 and an airfoil 78. The airfoil 78 extends from a tip 80 to a root 82. The dovetail 76 further includes a platform 84 integral with the airfoil 78 at the root 82, which helps to radially contain the turbine airflow. The dovetail 76 can be configured to mount to a turbine rotor disk on the engine 10. The dovetail 76 comprises at least one inlet passage, exemplarily shown as a first inlet passage 88, a second inlet passage 90, and a third inlet passage 92, each extending through the dovetail 76 to provide internal fluid communication with the airfoil 78 at a passage outlet 94. It should be appreciated that

the dovetail 76 is shown in cross-section, such that the inlet passages 88, 90, 92 are housed within the body of the dovetail 76.

Turning to FIG. 3, the airfoil 78, shown in cross-section, has a concave-shaped pressure sidewall 98 and a convex-shaped suction sidewall 100 which are joined together to define an airfoil shape with a leading edge 102 and a trailing edge 104. The airfoil 78 rotates in a direction such that the pressure sidewall 98 follows the suction sidewall 100. Thus, as shown in FIG. 3, the airfoil 78 would rotate upward toward the top of the page.

The airfoil 78 can comprise an interior 96 with a plurality of internal passages, illustrated by example as a first passage 106 and a second passage 108, separated by a rib 110, which can be arranged to form one or more cooling circuits dedicated to cool a particular portion of the airfoil 78. The passages 106, 108 can extend radially within the airfoil 78, from root-to-tip. It should be appreciated that the passages can comprise one or more film holes that can provide fluid communication between the particular passage and the external surface of the airfoil 78, providing a film of cooling fluid along the external surface of the airfoil 78.

In FIG. 4, a perspective view best illustrates the tip 80 of the airfoil 78. The pressure and suction sidewalls 98, 100 extend beyond the top surface of the tip 80 such that the top surface is defined as a tip wall 120 disposed between a pressure side extension 122 and a suction side extension 124. The combination of the tip wall 120, and the extensions 122, 124 can define a tip channel 121 disposed along the tip 80. A tip shelf 126 can be defined in the pressure side extension 122 as a groove located on the pressure sidewall 98. The airfoil 78 can further have one or more film holes 132 disposed therein, having the film holes 132 exemplarily illustrated on the pressure sidewall 98. Additionally, the airfoil 78 can have a plurality of exit apertures shown as slot channels 118 at the trailing edge 104, having a tip exit 116 disposed at the trailing edge adjacent the tip 80. Alternatively, a center extension (not shown) can extend from the tip wall 120 between the pressure and suction extensions 122, 124, dividing the tip channel 121 into separate tip channels.

Turning now to FIGS. 5A-5C, a cross-sectional view taken along section V of FIG. 4 illustrates a cooling passage as a tip passage 130, which can comprise the first or second passages 106, 108 of FIG. 3. FIG. 5A is a prior art tip 80 for the airfoil 78 having a film hole 132 extending between the tip passage 130 and the exterior of the airfoil 78 at the tip wall 120. The tip passage 130 can have either a symmetrical or an asymmetrical cross-section. The film holes 132 can be a compound film hole, having a first portion 134 and a second portion 136, which can define a metering section and a diffusing section, respectively. The compound angle of the film holes can be defined as having both an axial component and a radial component relative to the engine centerline 12. Alternatively, the compound film hole can be defined as having a span-wise component and a chord-wise component, relative to the span and the chord of the airfoil 78. Thus, it can be understood that although the film holes 132 are shown in cross-section being substantially radial, i.e. orthogonal to the engine centerline 12, the film holes 132 can also extend in an axial direction relative to the engine centerline 12, or a combination of axial and radial. Furthermore, the film holes 132 can be non-linear, defining at least a portion of an arcuate profile. The film holes 132 are not restricted to being compound. They can be axial, radial, linear, angled, compound, arcuate or otherwise in non-

limiting examples. Thus, a centerline of the film hole 132 can be straight, curved, arcuate, jointed, or otherwise.

A plurality of corners 138 can be defined at the junctions between the tip wall 120 and the pressure and suction wall extensions 122, 124. While the corners 138 are shown as sharp corners having a defined point, it should be appreciated that they are not so limited. For example, the corners 138 can be slightly rounded, or otherwise, such that a virtual corner can be defined. A corner angle 128 can be defined at each corner 138. The intersecting walls to define the corner 138 can intersect in a manner that defines an acute, right, or obtuse angle 128 for the corner 138. Additionally, one or more of the intersecting walls can be angled or arcuate, such that the corner 138 comprises an increasing cross-sectional distance extending from the corner 138 at the junction between the two walls.

Turning to FIG. 5B, the corner 138 adjacent the pressure side extension 122 and the tip wall 120 comprises a fillet 140. The fillet 140 defines an increased thickness at the corner 138 such that the length of the film hole 132 can be extended. In FIG. 5C, the fillets 140 can be disposed adjacent to either the pressure side extension 122, the suction side extension 124, or both. It should be further appreciated that nominally rounded corners are not equivalent to the fillets described herein. The fillets 140 define an increased thickness, while the slightly rounded corner can be nominal.

Turning now to FIG. 6, the fillet 140 as shown is exemplary and should be understood that the fillets 140 are a material filling the corner 138 at the junction of intersecting walls, defining an increased thickness. The geometries as shown in FIG. 6 should be understood as exemplary, and should not be construed as limiting of the invention. Alternatively, the fillet 140 could be defined as a compound fillet, having discrete arcuate or linear surfaces defining the fillet 140.

A length L can be defined as the length between an inlet 160 and an outlet 162 of the film hole 132 and a diameter D can be defined as the cross-sectional width of the film hole 132. The length can be determined as the distance between the inlet 160 and the outlet 162 through the center of the film hole 132 where the centerline intersects the airfoil surface. As such, the film hole 132 can be defined by the length-to-diameter ratio, L/D. While the film holes 132 are illustrated as having an increasing cross-sectional area at the second portion 136, it should be understood that the film holes 132 can comprise a consistent diameter D and cross-sectional area. Alternatively, the centerline of the film hole 132 can be straight, curved, arcuate, jointed, and any other suitable shape in non-limiting examples.

The suction sidewall 100 and the suction sidewall extension 124 can have a thickness 152 defined as a width for the wall. Similarly, the tip wall 120 can have a thickness 154. The increased thickness of the fillet 140 can be defined against the virtual extensions of the walls 120, 124, shown in dashed line, such that the extensions 120, 124 at the fillet 140 have an increased thickness being greater than the thicknesses 152, 154 of the respective walls 120, 124. It should be understood that the fillets 140 need not comprise additional material, but can be integral with at least one of the walls 120, 124 and can define a thickness against the virtual extensions of the walls 120, 124 shown in dashed lines adjacent the fillet 140. It should be further appreciated that the virtual extensions of the walls are for illustrative purposes, providing the reader with a means of visual comparison of the thickness of the fillet 140 against the wall

thicknesses **120**, **124**, and such an extension of the walls adjacent the fillet **140** are not required.

The fillet **140** can define at least a portion of a circle or an arcuate surface **150**, such that an effective radius **158** is defined between the fillet **140** and a center point **156** of the arcuate surface **150**. The fillet **140** comprises a thickness extending between the corner **138**, shown in dashed line, and the arcuate surface **150**. The fillet **140** is shaped such that the effective radius **158** is at least 1.5 times larger than the greater of the first or second thickness **152**, **154**. Alternatively, the shape and size of the fillet **140** can be adapted to increase the length of the film holes **132**. Increasing the length L of the film holes **132** increases the value for the L/D ratio for the film holes **132**. Furthermore, the angle **128** (FIG. 5A) defined by the corner **138** can provide for an increased or decreased effective radius **158**. For example, the filleted corner **138**, as illustrated, is a right angle providing for an effective radius **158** of approximately 2.0-2.5 times the larger of the thicknesses **152**, **154**. In alternative corners **138** where the angle **128** is acute or obtuse, for example, the fillet **140** can define an effective radius, which can be greater or smaller than the exemplary embodiment shown. As such, the fillet **140** can define an effective radius of 1.5 to 10.0 times the thicknesses **152**, **154**, or more. It should be understood, however, that the thickness defined by the fillet **140** is not limited to being defined by the angle **128** of the intersecting walls **120**, **124**.

While the thicknesses **152**, **154** are shown in reference to the tip wall **120** and the suction sidewall extension **124**, it should be appreciated that the respective geometries of the tip passage **130** within the airfoil **78** as shown is exemplary and should not limit the airfoil **78** to the geometries, dimensions, proportions, or positions as shown. The fillet **140** can be defined in additional places at the tip **80** of the airfoil **78** and will be fully described in the examples shown in FIGS. 7-10.

It should be further appreciated that the circle shape defined by the fillet **140** is exemplary. The fillet **140** need not be shaped such that fillet **140** defines the circle shape. The fillet **140** can be any arcuate shape or segment thereof, such that a radius or local radius can define the effective radius **158**. The fillet **140** can be a non-circular arc, such that a segment of the arc or at least a portion of the fillet **140** can define a local radius to comprise the effective radius **158**. Alternatively, the fillet **140** could be defined as a compound fillet with a compound radius of curvature, having discrete arcuate or linear surfaces defining the fillet **140**. When utilizing a compound radius of curvature defined by the fillet **140**, the average overall radius could be used to determine the effective radius **158**. Further still, the fillets **140** accommodate film hole inlet or exit shaping, as well as non-linear geometries. As such, exit shaping should be understood as any shape other than a standard round inlet or exit.

Turning now to FIGS. 7A-9B, multiple examples for implementation of a fillet **140** near the tip **80** are shown. In FIGS. 7A-7C, the fillet **140** can be disposed within the tip passage **130** of the airfoil, illustrating the cross-section VII of FIG. 4. Looking at FIG. 7A, showing a prior art blade tip, the corners **138** can be defined internal of the airfoil **78**, adjacent to the tip wall **120** and the pressure and suction sidewalls **98**, **100**. In FIG. 7B, the fillet **140** is disposed adjacent to the pressure sidewall **98**, providing for an increased length for the film hole **132**, such that the L/D ratio defined by the film hole **132** can be increased. FIG. 7C shows an additional exemplary embodiment having a fillet **140** adjacent to both the pressure and suction sidewalls **98**,

100. Thus, it should be understood that the fillet can be disposed adjacent to the pressure sidewall **98**, the suction sidewall **100**, or both.

Turning now to FIGS. 8A-8C, the tip **80** of the airfoil can be defined by section VIII of FIG. 4, having film holes **132** extending from the tip passage **130** to the pressure sidewall **98** of the airfoil **78**. In FIG. 8A illustrating a prior art blade tip, similar to FIG. 7A, the corners **138** are defined adjacent to the tip wall **120** and the pressure and suction sidewalls **98**, **100**. The film hole **132** extends through the pressure sidewall **98** providing fluid communication between the tip passage **130** and the side of the airfoil **78**.

In FIG. 8B, the corner **138** adjacent to the pressure sidewall **98** comprises the fillet **140**, providing for an increased L/D ratio for the film hole **132** extending to the external surface of the pressure sidewall **98**. In FIG. 8C, another example illustrates the potential to have both a film hole **132** disposed on the pressure and suction sidewalls **98**, **100**. Thus, it should be understood that the film hole **132** can be disposed through the fillet **140** to either the pressure sidewall **98**, the suction sidewall **100**, or both.

Turning now to FIGS. 9A-9C, another example illustrates fillets **140** being disposed within a serpentine section of an internal cooling circuit. Looking at FIG. 9A, showing a prior art turn **170** of a cooling circuit as the tip passage, the corners **138** can be defined at the junction between the tip wall **120** and one or more internal ribs **172**, extending along the radial, span-wise length of the airfoil **78**. Turning to FIGS. 9B and 9C, the fillets **140** can be placed at the corners defined against the internal ribs **172** at the tip wall **120**. The fillets **140** can be utilized to increase the length of the film holes **132** defining a greater L/D ratio for the film holes to increase film cooling effectiveness at the tip **80**.

Turning to FIGS. 10A-10B, taken at the section IX of FIG. 4, the tip **80** of the airfoil **78** with the tip shelf **126** disposed in the pressure sidewall **98**. In FIG. 10A, similar to FIGS. 7 and 8, the corners can be disposed within the tip passage **130** adjacent to the tip wall **120** and the pressure and suction sidewalls **98**, **100**. The film hole **132** is disposed in the tip wall **120** and extends to the tip shelf **126**. In FIG. 10B, the corner **138** adjacent to the pressure sidewall **98** at the tip shelf **126** can comprise a fillet **140**, providing for an increased L/D ratio for the film hole **132** extending to the tip shelf **126**. Alternatively, it is contemplated that the fillet **140** can be between the tip wall **120** and the side extension **122** at the tip shelf **126**, being external of the airfoil and located within the tip channel **121**.

The film hole **132** at the tip shelf **126** can be a shaped film hole or comprise a compound film hole as described herein. The film hole **132** can be disposed on any surface of the tip shelf **126**, such as a fillet, bottom shelf surface, radial face, or any combination thereof. Additionally, the film hole **132** can have any orientation, being radial, axial, tangential, or any combination thereof. The film hole **132** can extend from the filleted surface **140** to the tip shelf **126** or from the tip wall **120** through a portion of the filleted surface **140**, being interior of or exterior of the airfoil, to the tip shelf **126**. The fillet **140** provides for an increased length for the film hole **132**, defining a greater L/D ratio for the film hole **132** to improve film effectiveness.

It should be appreciated that as described herein, the filleted surfaces, being internal or external, increase the length for the film holes and the L/D values for film holes by locally increasing both the internal and external fillet radius through which the hole penetrates. The increased values for L/D provide for increased cooling film hole effectiveness. The fillets can minimally increase overall system weight

without thickening an entire wall or surface. Furthermore, the fillets provide for an increase in structural support. Further still, the fillets accommodate film hole inlet or exit shaping, as well as non-linear geometries. As such, exit shaping should be understood as any shape other than a standard round inlet or exit. Additionally, as described herein, any fillets can be used in combination with one another, such that two fillets are used to increase the L/D ratio for film holes extending through the fillets.

The film holes can be in a wall surface or a fillet surface and penetrate through at least a portion of the fillet, permitting an increased length for the film hole otherwise impossible to achieve without the fillet. Furthermore, the fillets can be compounded, such as filleted internal surfaces or the combination in external and internal surfaces to further increase the length to achieve greater L/D values.

It should be further appreciated that the fillets provide for an increased length providing the potential for a wider range of film holes, such as curved film holes, "S-curved" film holes as well as other orientations beyond a standard straight or compound film hole with increased effectiveness.

It should be further understood that while the embodiments as described herein relate to an airfoil, the filleted corners can be utilized in additional engine components having intersecting walls to define a corner and utilizing film holes or cooling at or near those corners of the engine components.

This written description uses examples to disclose the invention, including the best mode, and to enable any person skilled in the art to practice the invention, including making and using any devices or systems and performing any incorporated methods. The patentable scope of the invention is defined by the claims, and can include other examples that occur to those skilled in the art. Such other examples are intended to be within the scope of the claims if they have structural elements that do not differ from the literal language of the claims, or if they include equivalent structural elements with insubstantial differences from the literal languages of the claims.

What is claimed is:

1. A blade for a gas turbine engine comprising:
 - an airfoil having an outer wall defining a pressure side and a suction side, the outer wall extending chord-wise from a leading edge to a trailing edge and span-wise from a root toward a tip;
 - a tip wall spanning the pressure side and the suction side of the outer wall and intersecting the outer wall to form at least one corner, with the outer wall having a first thickness at the corner and the tip wall having a second thickness at the corner;
 - a cooling passage having a portion located along the tip wall and at least partially defined by the tip wall and the outer wall;
 - a fillet located at the corner and having an effective radius of at least 1.5 times larger than the greater of the first and second thicknesses; and
 - at least one film hole extending through the fillet to fluidly couple the cooling passage to an exterior of the airfoil.
2. The blade according to claim 1 wherein the at least one film hole includes an outlet provided on at least one of the pressure side or the suction side.
3. The blade according to claim 1 wherein the tip wall at least partially defines a tip channel between extensions of the pressure side and the suction side of the outer wall, and the at least one film hole extends to the tip channel.
4. The blade of claim 3 further including a second fillet provided at a second corner defined at a junction between

one extension of the pressure side or the suction side of the outer wall, with the at least one film hole extending through the second fillet.

5. The blade according to claim 1 further comprising a tip shelf disposed within the outer wall wherein the at least one film hole includes an outlet at the tip shelf.

6. The blade according to claim 5 further comprising a third fillet located at the tip shelf, with the at least one film hole passing through the third fillet.

7. The blade according to claim 1 wherein the corner extends chord-wise along the tip wall, the fillet extends chord-wise along the corner, and the at least one film hole comprises multiple film holes.

8. The blade according to claim 7 wherein the fillet is continuous.

9. The blade according to claim 7 wherein the fillet is discontinuous.

10. The blade according to claim 1 wherein the effective radius is at least 2.0 times larger than the greater of the first and second thicknesses.

11. The blade according to claim 1 wherein the effective radius is at least 4.0 times larger than the greater of the first and second thicknesses.

12. The blade according to claim 11 wherein the effective radius is less than 10.0 times larger than the greatest of the first and second thicknesses.

13. The blade according to claim 1 wherein the cooling passage has one to three corners.

14. The blade according to claim 1 wherein the cooling passage has one to two corners.

15. The blade according to claim 1 wherein the cooling passage has a symmetrical cross section.

16. The blade according to claim 1 wherein the cooling passage has an asymmetrical cross section.

17. The blade according to claim 1 wherein the film hole is linear.

18. The blade according to claim 1 wherein the film hole is non-linear.

19. The blade according to claim 1 wherein the film hole defines a centerline forming a compound angle.

20. The blade according to claim 19 wherein the compound angle comprises both a span-wise component and a chord-wise component.

21. The blade according to claim 20 wherein the film hole is non-linear.

22. The blade according to claim 21 wherein the film hole is curvilinear.

23. A method of forming a film hole in a blade of a gas turbine engine comprising forming the film hole through a fillet formed at a corner formed by an intersection of a tip wall and an outer wall, with the fillet provided in an interior of the blade and with the film hole having an outlet provided on the tip wall; wherein an effective radius of the fillet is at least 1.5 times larger than the greater of a thickness of the tip wall or a thickness of the outer wall.

24. The method of claim 23 wherein the effective radius of the fillet is greater than 2.0 times the thickness of the greater of the thickness of the tip wall or the thickness of the outer wall.

25. The method of claim 24 wherein the effective radius is at least 4.0 times larger than the greater of the thicknesses among the thickness of the tip wall or the thickness of the outer wall.

26. The method of claim 24 wherein the effective radius is less than 10.0 times the thickness of the greater thickness among the thickness of the tip wall and the thickness of the outer wall.

27. A blade for a gas turbine engine comprising:
an airfoil having an outer wall defining an interior separate from an exterior, and including a pressure side and a suction side, the outer wall extending chord-wise from a leading edge to a trailing edge and span-wise from a root toward a tip;
a tip wall spanning the pressure side and the suction side of the outer wall and intersecting the outer wall to form a first interior corner at the intersection of the tip wall and the pressure side and a second interior corner at the intersection of the tip wall and the suction side;
a cooling passage at least partially defining the interior;
a first fillet located at one of the first interior corner or the second interior corner;
a first film hole extending through the first fillet;
a second fillet located at the other of the first interior corner or the second interior corner; and
a second film hole extending through the second fillet;
wherein at least one of the first film hole or the second film hole extends through the tip wall; and
wherein the first fillet and the second fillet each include an effective radius of at least 1.5 times larger than the greater of a thickness of the tip wall or a thickness of the outer wall.

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25