

996,814.

E. J. VON PEIN.
CASH REGISTER.
APPLICATION FILED OCT. 2, 1909.

Patented July 4, 1911.

4 SHEETS—SHEET 1.

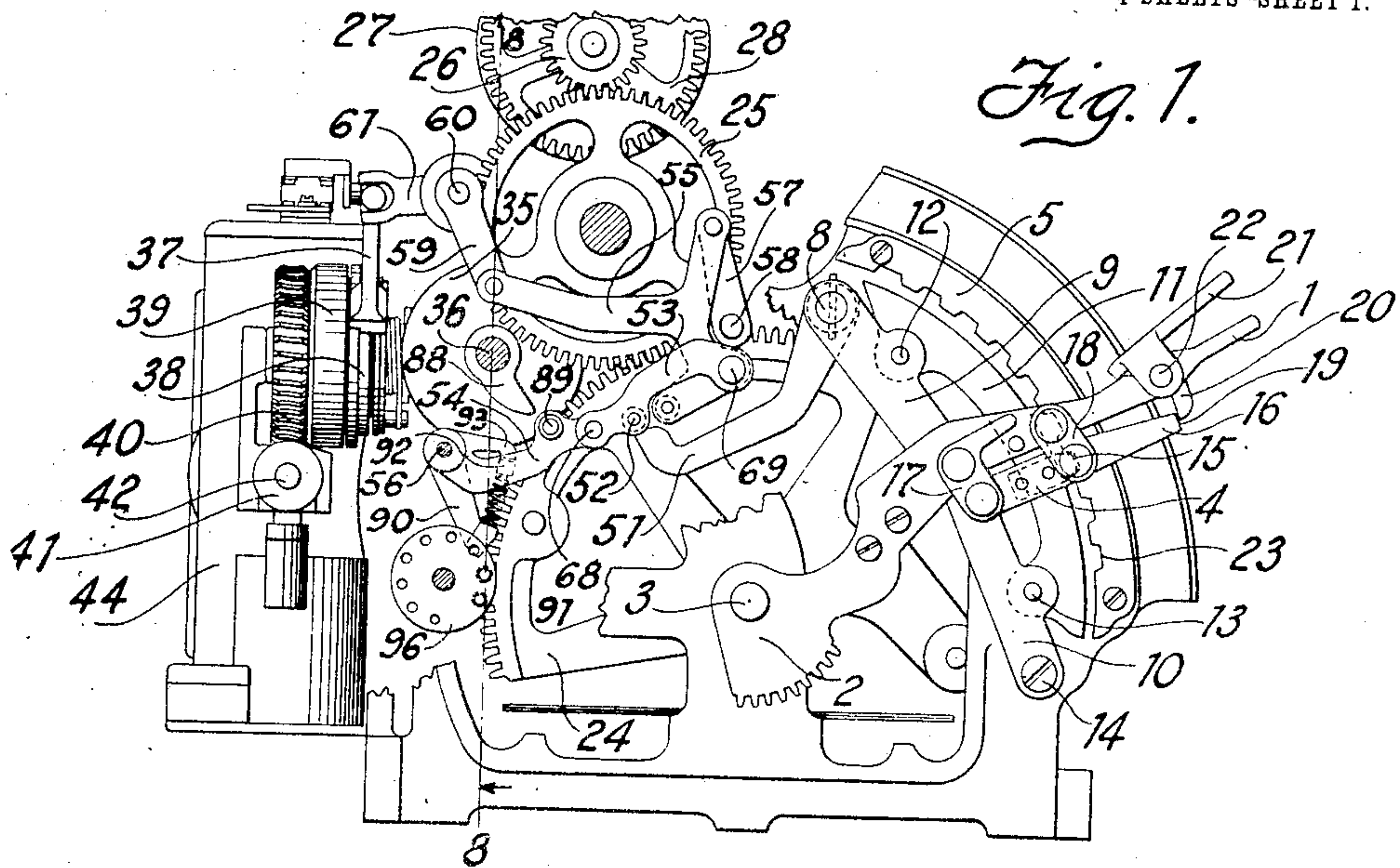


Fig. 1.

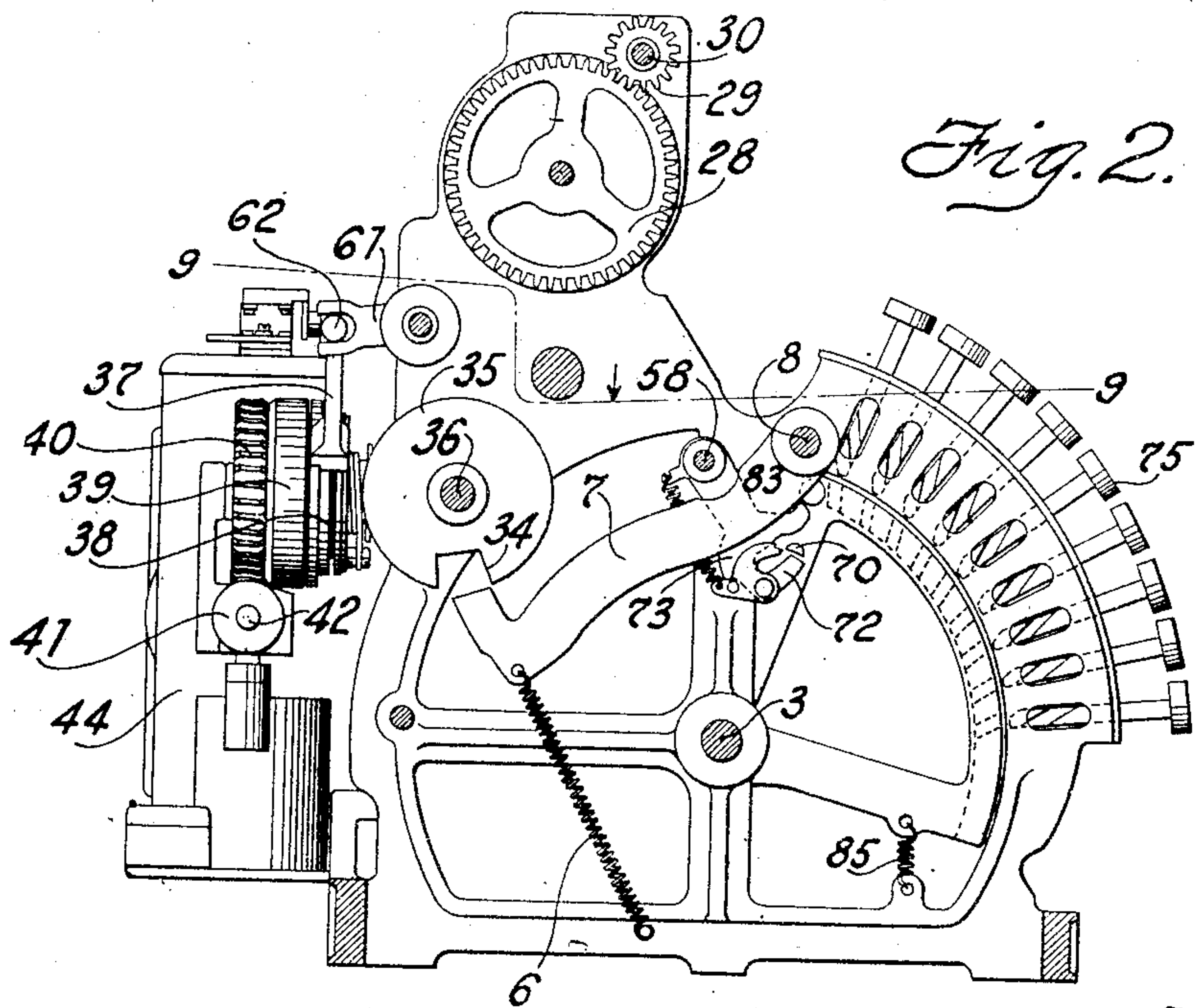


Fig. 2.

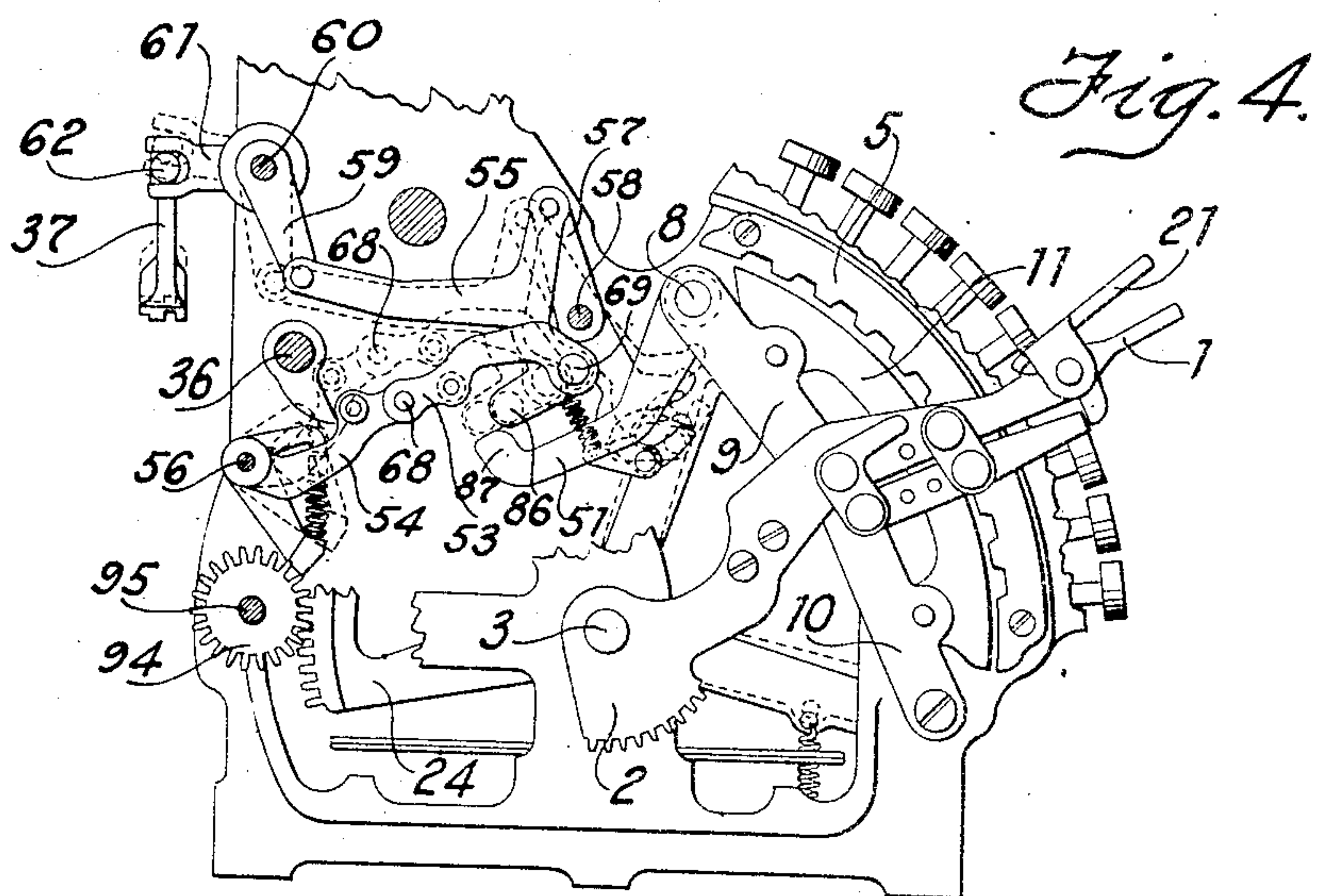
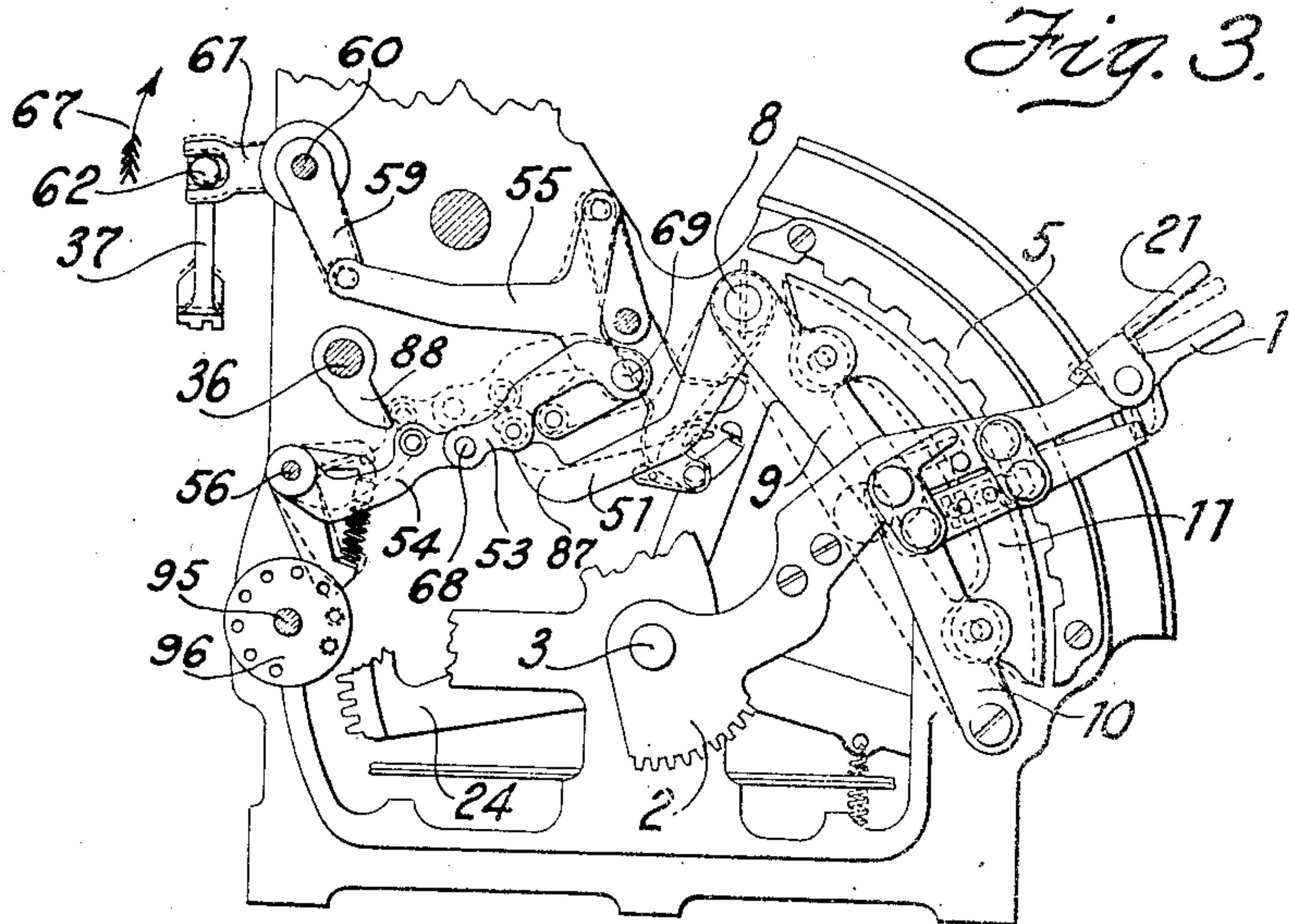
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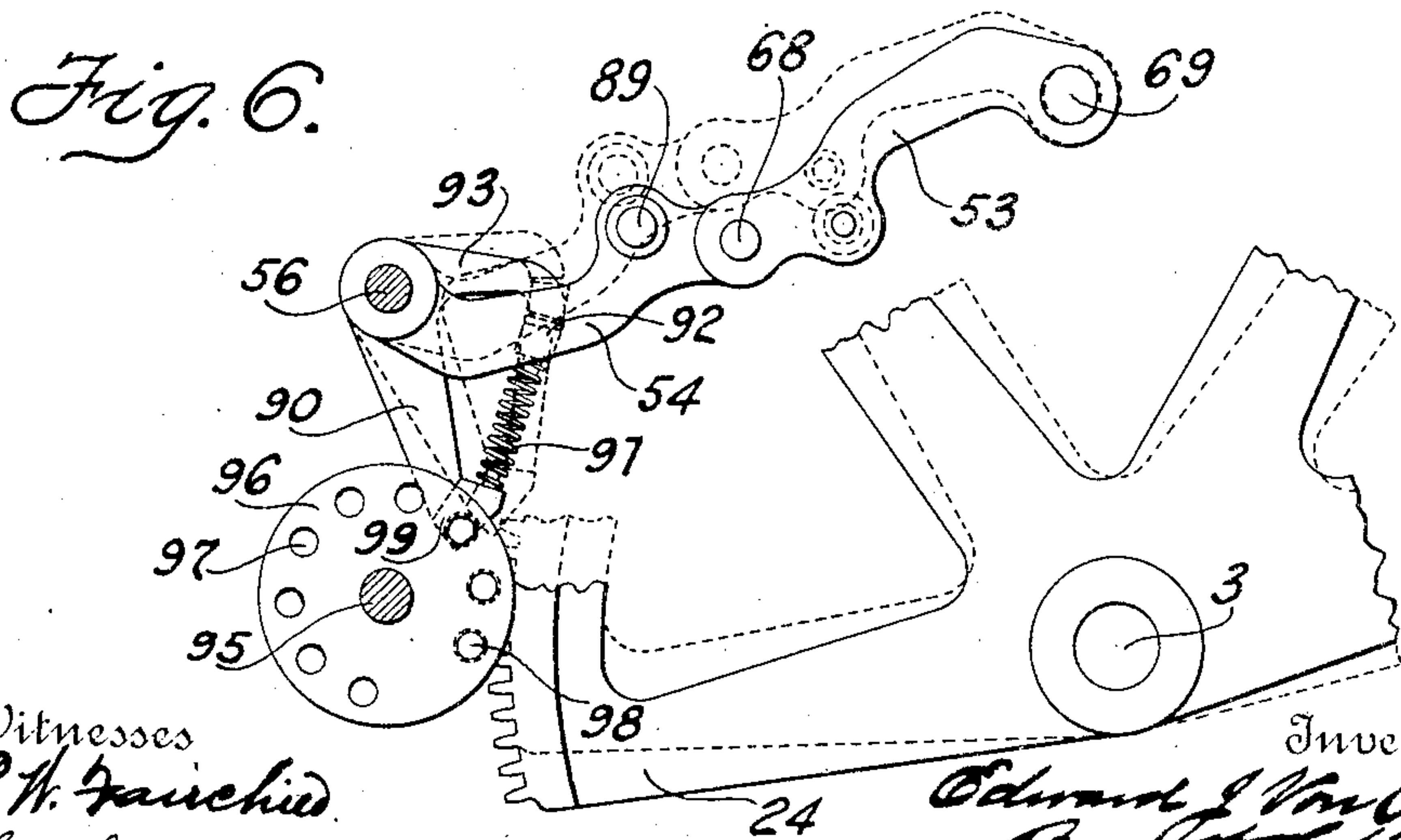
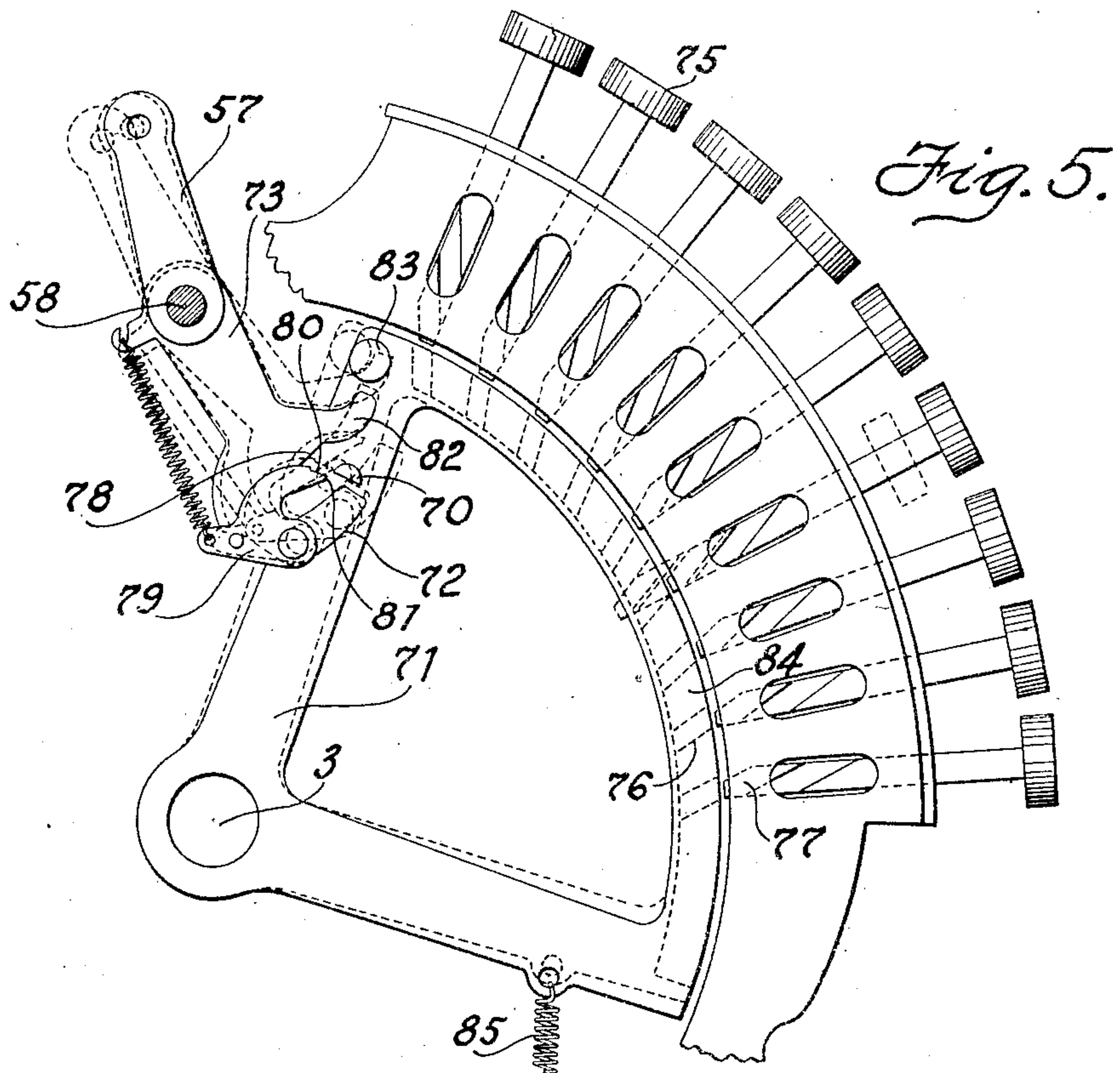
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4 SHEETS—SHEET 3.



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4 SHEETS-SHEET 4.

Fig. 7.

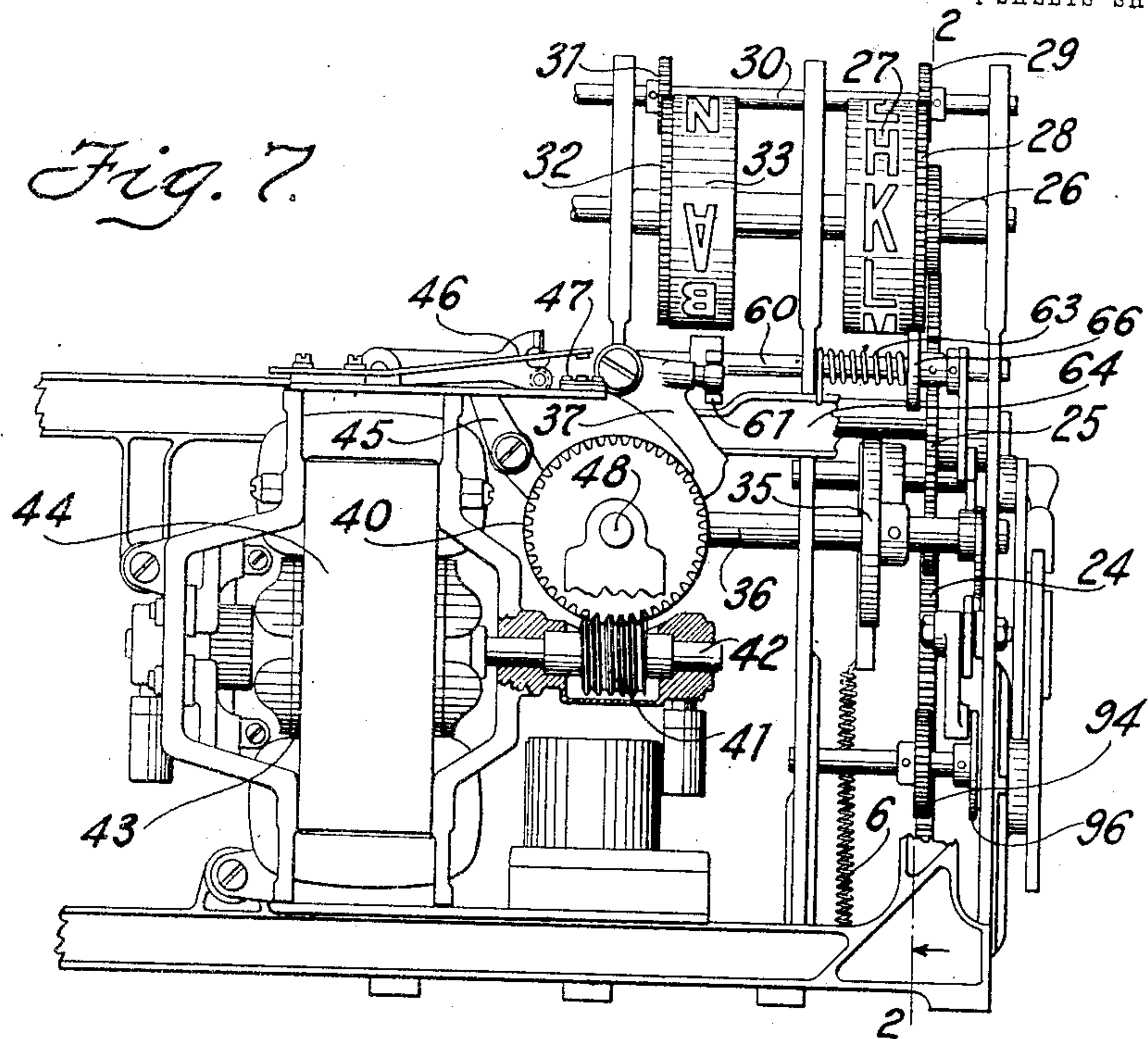
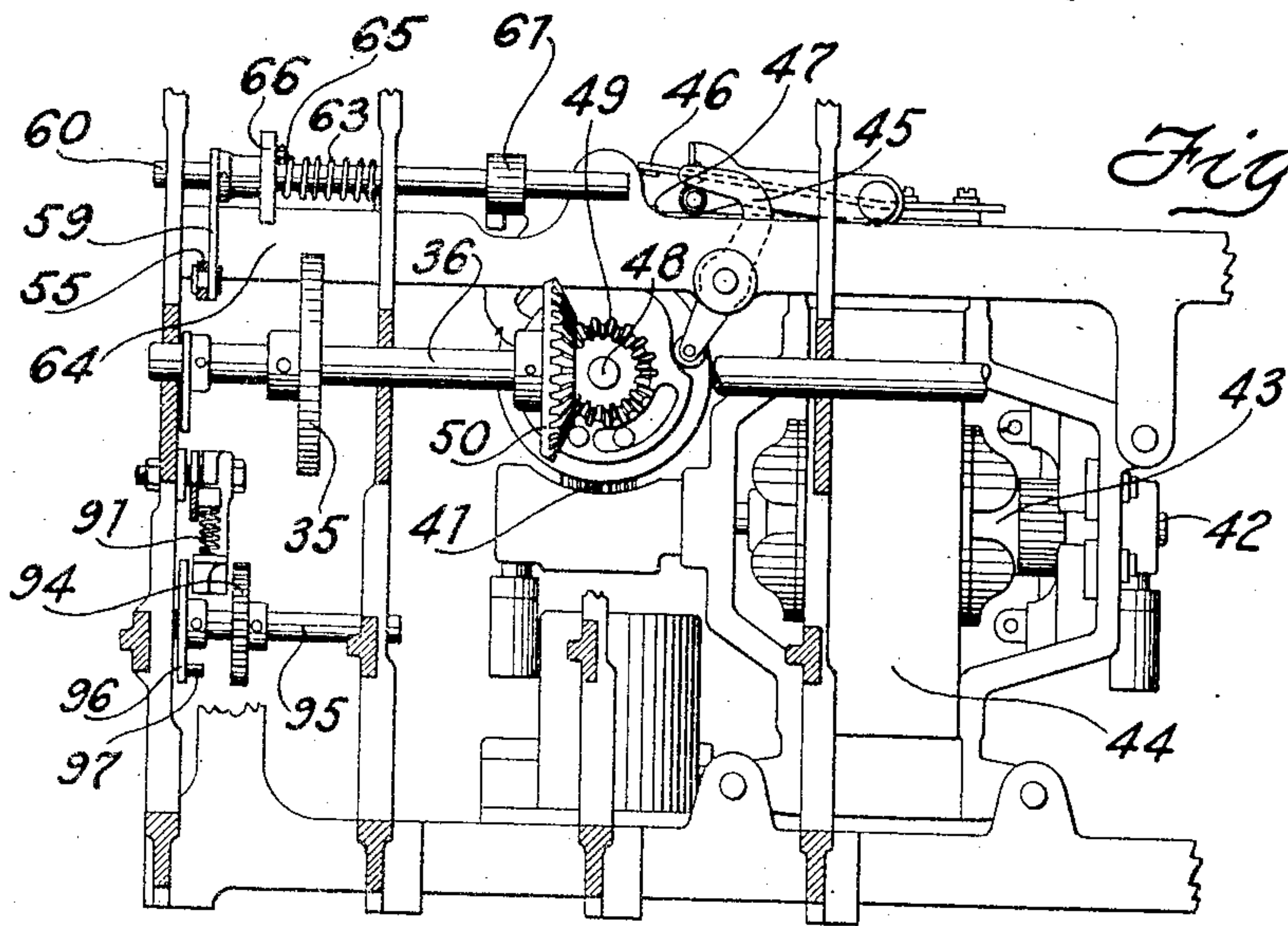


Fig. 8.



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UNITED STATES PATENT OFFICE.

EDWARD J. VON PEIN, OF DAYTON, OHIO, ASSIGNOR TO THE NATIONAL CASH REGISTER COMPANY, OF DAYTON, OHIO, A CORPORATION OF OHIO, (INCORPORATED IN 1906.)

CASH-REGISTER.

996,814.

Specification of Letters Patent.

Patented July 4, 1911.

Application filed October 2, 1909. Serial No. 520,653.

To all whom it may concern:

Be it known that I, EDWARD J. VON PEIN, a citizen of the United States, residing at Dayton, in the county of Montgomery and State of Ohio, have invented certain new and useful Improvements in Cash-Registers, of which I declare the following to be a full, clear, and exact description.

This invention relates to multiple totalizer cash registers, of the type which are provided with manipulative means for bringing any desired totalizer into operative relation with the totalizer actuators.

The manipulative means has heretofore coacted in such a way with machine locking devices that it was necessary to operate the manipulative means before each operation of the machine, for the purpose of compelling an operation of this manipulative means as a reminder to the operator so that he would not forget to set the same and thereby possibly make the record in a wrong totalizer. Very often some one or more totalizers are operated much more frequently than others and under such circumstances it is desirable not to compel successive operations of the manipulative device when such a one of the totalizers is to be repeatedly in operative relation with the actuating mechanism.

The present invention relates to and has for its main object to provide adjustable means which may be arranged to act in such a way that upon any desired positions of the manipulative device it will be unnecessary to first operate the same in order to operate the machine.

With this and incidental objects in view the invention consists in certain novel features of construction and combinations of parts, the essential elements of which are set forth in appended claims and a preferred form of embodiment of which is hereinafter described with reference to the drawings which accompany and form part of the specification.

Of said drawings: Figure 1 is a left side elevation of a cash register constructed according to this invention, but with the multiple totalizers omitted. Fig. 2 is a transverse vertical section taken on the line 2—2 of Fig. 7. Fig. 3 is a right side elevation similar to Fig. 1, but showing some of the mechanism both in its normal position and

in an intermediate position between its normal position and its fully moved position. Fig. 4 is a view similar to Fig. 3, but showing part of the mechanism both in its normal position and its fully moved position. Fig. 5 is a detail of the clerks' initial bank of keys. Fig. 6 is a detail of the setting lever throwout mechanism. Fig. 7 is a rear elevation of the register partly broken away. Fig. 8 is a longitudinal section taken on the line 8—8 of Fig. 1.

The drawings show a pivoted lever serving as the manipulative device by which different totalizers may be brought into operative relation with the actuating mechanism. This lever carries a segment gear which is adapted to mesh with a pinion which is fixed to and concentric with a reel for carrying the totalizers. The specific arrangement of the totalizers and their actuating mechanism and the means for controlling the actuating mechanism is fully illustrated and described in the patent to Thomas Carroll #703,639, granted July 1, 1902.

The setting lever is indicated in the drawings at 1, the segment gear above referred to which is part of the setting lever is indicated at 2, and the pivot of the lever at 3. By rocking the lever 1 on its pivot different degrees the segment gear 2 causes the clerks' totalizer reel to be rotated different degrees thereby bringing different totalizers into operative relation with the actuating mechanism.

The lever 1 is alined in its different settings by a plunger 4 which co-acts with a plate 5 which is suitably curved for the purpose and which has notches into which the plunger moves. The plunger 4 is spring pressed toward the plate 5 by a spring 6 (Fig. 2). The spring 6 normally draws a locking arm 7 downwardly, this arm being fixed to a short shaft 8 which shaft also carries one arm 9 of a pair of arms 9 and 10 which carry the plate 11 at the pivots 12 and 13. The arm 10 is pivoted to the frame of the machine on a pin 14. As the locking arm 7 is normally drawn downwardly, the plate 11 will be forced toward the alining plate 5. The plunger 4 is provided with a shoulder 15 extending across the front edge of the plate 11, and consequently is urged into engagement with the alining plate 5.

The plunger 4 is carried by a bar 16 which is hung from the lever 1 by links 17 and 18 and at 19 coacts with a downwardly extending lug 20 of a thumb grip 21 which is pivoted to the lever 1 on a pin 22. By forcing the thumb grip 21 toward the lever 1, the lug 20 engages the bar 16 thus moving the plunger 4 out of engagement with the alining bar 5. The lever 1 may then be set to any of the nine positions determined by the nine notches 23.

The lever 1 is rigid with a large segment gear 24 which meshes with an intermediate gear 25 which in turn meshes with a pinion 26 fixed to an indicator drum 27. This drum is provided with nine characters representing the different totalizers and the settings of the lever 1. The character which is brought opposite to a suitable sight opening in the cabinet at the rear of the register indicates which totalizer is in operative position. The drum 27 carries a gear 28 (Fig. 7) meshing with a pinion 29 on the shaft 30 which shaft carries a similar pinion 31 meshing with a gear 32 fast to a second indicator drum 33. The drum 33 is similar to the drum 27 but the indicating characters are reversed on it for the purpose of indicating at the front of the register. By forcing the thumb grip 21 toward the lever 1, the locking arm 7 is forced into the cut 34 of a disk 35 which is carried by the main drive shaft 36 of the machine. Because of this arrangement it is impossible to operate the machine except when the plunger 4 is in one of the notches in the alining plate 5.

The machine is normally locked against operation regardless of the position of the locking arm 7 by an arm 37 (Figs. 1 and 2) which is normally in the path of a shoulder of a disk 38 which is carried by one member of a clutch 39. The co-acting member of the clutch is carried by a spiral gear 40. This gear meshes with a worm 41 which is carried by a shaft 42 fast to the armature 43 of a motor 44. The specific construction of the clutch 34 is described and illustrated in an application filed by C. F. Kettering and W. A. Chryst, Serial No. 394,187, filed September 23, 1907. As described in said application the disk 38 has a relative movement with respect to the member of the clutch which carries the disk for the purpose of actuating an arm 45 (Fig. 7) which at such time forces contacts 46 against a contact 47. These contacts are suitably connected to the source of electrical energy and to the motor 44 so that when the same are closed the spiral gear 40 will be rotated. This gear through the clutch 39 transmits energy from the motor to a stub shaft 48 (Figs. 7 and 8), which carries a bevel gear 49 meshing with a bevel gear 50 on the drive shaft 36. Because of this arrange-

ment, when the arm 37 is rocked upwardly, thereby permitting the rotation of the shaft 48 and closing the contacts 46 and 47, the main drive shaft 36 will be rotated. The arrangement is such that the shaft 36 is permitted to make one rotation only, upon each operation of the machine. The contacts 46 and 47 are automatically opened at the proper time by mechanism which is described in the said application. The manner of returning the arm 37 into the path of the disk 38 so that the drive shaft 36 will be stopped at the completion of a rotation, will be stated in the ensuing description.

The mechanism is arranged so that the locking arm 37 may be released by operation of the thumb grip 21. To this end the arm 37 carries a roll 62 and surrounding this roll is the fork of an arm 61 mounted fast on a short shaft 60 to which a downwardly extending arm 59 is also fast. A link 55 connects the arm 59 to a second approximately parallel arm 57 fast on a shaft 58. A pair of toggle arms 53 and 54, connected by a pin 68, are provided, and toggle arm 53 is connected to link 55 by a pin 69 while toggle arm 54 is fast on a stub shaft 56 which is mounted in the machine frame. In the normal position of the parts as shown in Fig. 1 the line joining pivots 56 and 69 of the toggle arms passes above the connecting pin 68 and therefore the locking arm 37 is held fast in locking position, but if the toggle is broken by elevation of pin 68, the locking arm may then be moved to releasing position by further rotation of the toggle arms, and in fact would be so moved by a spring 63 (Fig. 8) surrounding and fastened to the shaft 60, except for other parts yet to be mentioned.

In order to break the toggle by operation of thumb grip 21, an arm 51 is provided fast on shaft 8. Thus when the grip 21 is operated plunger 4 will be depressed, and will through plate 11 and arm 9 rock this shaft thereby raising the rear end of arm 51 to engage and move a roller 52 carried by toggle arm 53. As soon as the connecting pin 68 passes above the line between pivot pins 69 and 56 the spring 63 becomes effective and tends to move locking arm 37 to its releasing position. The action of the spring would, in this case, result in rocking the shaft 60 in the direction indicated by the arrow 67 and thereby cause the arm 37 to release the clutch mechanism, but the shaft 60 cannot rock unless the arm 59 which it carries is also permitted to rock and draw the link 55 rearwardly bringing the pivot 69 toward the pivot 56, and advantage is taken of this to compel a depression of one of a bank of keys before the machine may be finally released. Such action is normally prevented by a pin 70 (Fig. 5) which is carried by a key detent 71 and is normally

in the path of the shoulder 72 of an arm 73 carried by the shaft 58 which shaft also carries the arm 57 from which is hung the link 55.

5 The key detent 71 may be actuated to carry the pin 70 inwardly clear of the shoulder 72, by any one of the keys 75 of a bank of special keys which are illustrated in Fig. 5. This bank of special keys controls elements of indicating mechanism and printing mechanism and generally special counters as is customary in cash registers. As the devices which are controlled by the special keys relate indirectly only to the present invention, they are not illustrated in the drawings, but as this bank controls mechanism for the purpose of recording the nature or class of every transaction which is recorded in the machine, the special bank is arranged to co-act with the machine locking mechanism so that it is impossible to operate the machine until one of the special keys has been depressed. For this reason the key detent 71 which is moved upwardly upon the depression of a key is provided with the stop pin 70 which before the detent is rocked upwardly is in the path of the shoulder 72 and consequently prevents the actuation of the spring 63. The detent 71 is loosely pivoted upon the shaft 3 and is provided opposite the keys with grooves 76 which are inclined to the path of movement of the keys. The shanks of the keys are provided with coacting inclined shoulders 77 and when any one key is depressed its shoulder 77 enters one of the grooves 76 causing the detent to be rocked upwardly so that the pin 70 will move into the slot 78 of the arm 72. When the pin 70 is out of the path of the shoulder 72, the spring 63 rocks the stop arm 37 away from the disk 38 thereby causing the contacts 46 and 47 to be closed and the operation of the motor, which results in the rotation of the drive shaft 36. When the pin 70 is clear of the shoulder 72 the shaft 60 under the action of the spring rocks to carry the stop arm 37, the forked arm 61 and the arm 59 from the position in which they are shown in full lines in Fig. 4 to the position in which they are shown in broken lines in the same figure. This movement of the arm 59 draws the link 55 rearwardly thus moving the members 53 and 54 of the toggle as indicated by the broken lines in Fig. 4. The movement of the link 55 also rocks its carrying arm 57 and the shaft 74 causing the arm 73 which the shaft carries to rock upwardly, with the pin 70 in the inner end of the slot 78 in the arm 73.

60 It will be apparent after examining the structure as indicated by the drawings and so far described that in order to cause the machine to operate as desired it is necessary to actuate the lever 1 before the special key is depressed. If it was possible to actuate

the machine by first depressing the special key and then setting the lever 1, upon forcing the thumb grip toward the lever 1, the stop arm would swing causing the machine to operate. As this would happen before the lever could be moved to a new position means are provided which prevents the operation of the machine unless the lever 1 is set before the special key is depressed. This means consists of a pawl 79 which is carried by the arm 73 and is normally in such position that it will protrude over the slot 78 in the lever 73 and therefore prevent the lever 73 from rocking upwardly as the pin 70 would then engage the surface 80 of the pawl. The pawl 79 is normally below the path of movement of the pin 70 when the detent is rocked backwardly upon the depression of one of the keys, and if a key is depressed before the lever 1 is actuated the pin 70 will pass over the surface 80 of the pawl as the detent moves rearwardly. Therefore if the lever 1 is operated after the depression of the key, the machine will not operate.

If the lever 1 is operated before the depression of the special key, the initial movement of the arm 73 will cause the tip 81 of the pawl 79 to come into the path of movement of the pin 70. Then upon the depression of a special key the pawl is rocked out of the path of the pin allowing the lever to rock its full distance upwardly thus permitting the operation of the machine. In such case the shoulders 72 and 82 rock in front of the pin 70 and roller 83 preventing the return of the detent 71 until the arm 73 is restored to its initial position. When the detent is thus locked in its moved position it is impossible to return the depressed key and it is also impossible to depress another in the same bank as the uncut portions 84 of the detent are then in the paths of the keys. The detent is drawn to its normal position after the return of the arm 73 by the spring 85. During the rocking of arm 73 the roller 83 relieves the pin 70 from friction and wear which otherwise would occur by the pin 70 rubbing against the shoulder 72.

As the retaining of the detent 71 in its moved position depends upon the actuation of the arm 73, if a key 75 is depressed before the lever 1 is operated, upon the release of the key it will be immediately returned to its undepressed position through the action of a key spring or of the spring 85 on the detent 71. In such case it will be necessary again to depress the special key after the lever 1 is operated.

If the special key is depressed and held in its depressed position and the lever 1 then actuated, the arm 73 will endeavor to rock forwardly, but will be prevented by the pawl 79 engaging the pin 70. It will then be

necessary to release the key and again depress it in order to rock the pawl out of stopping relation with the pin 70.

The link 55 carries a roll 86 which, it will be seen by referring to Fig. 4, moves into engagement with the end 87 of the arm 51, upon the backward movement of the link 55 when the locking arm 37 is moved away from the disk 38. Because of the roll 86 being in the path of the arm 51 it is impossible to operate the lever 1 while the driving mechanism of the machine is in motion as the arm 51 is fixed to the shaft 8 which carries the arm 9 which is one of the carrying arms of the plate 11. The forward edge of this plate extends across the rear side of the lug 15. It is therefore impossible to move the plunger 4 out of engagement with the alining plate 5 except by forcing the plate 11 rearwardly and consequently rocking the arm 51 upwardly.

The drive shaft 36 carries a camming arm 88 which when the drive shaft nears its home position forces a roller 89 on the arm 54 of the toggle to its original position, thereby returning the arm 73 (Fig. 5) to its original position permitting the detent 71 to return to normal position and so releasing the key, and also causing the return of the stop 37 into the cut away part of the disk 38 so as to engage the shoulder of the disk and stop the disk when it arrives at its home position. The contacts 46 and 47 are also opened when the shaft 36 reaches its home position.

From the foregoing description it will be understood that it is impossible to operate the machine except by first actuating the lever 1 and therefore actuating the toggle composed of the arms 53 and 54 so that the spring 63 may cause the locking arm 37 to be moved away from the disk 38. The releasing mechanism thus far referred to is to be found in said Kettering and Chryst application and the description has been repeated herein merely for the purpose of making the operation of the parts clear, so that the present invention, which will next be described, may be readily understood.

At times certain totalizers are used much more frequently than others and for this reason when the lever is set to the position for adding upon such totalizers there is no need of requiring an operation of the lever 1 merely to call the operator's attention to whether or not the correct totalizer is in mesh. For the above reason adjustable means are provided which may be set so that upon certain positions of the lever 1 it will be unnecessary to first operate the same before operating the machine. This means comprises a device for automatically returning the toggle arms 53 and 54 upwardly so that the pivot 68 will be above the plane passing through the centers of the shafts 56

and 69. The cam 88 on the drive shaft 36 as explained rocks the arms 53 and 54 downwardly at the completion of each operation. The shaft 56 which carries the arm 54 has loosely pivoted thereon an arm 90 (Fig. 6) and a spring 91 is interposed between the arm 90 and a lug 92 on the arm 54. The arm 90 also has an extension 93 which engages the upper surface of the lug 92 for the purpose of preventing the spring 91 from rocking the arm 90 away from the arm 54 except to a limited extent. It may be seen because of the spring connection between the arms 54 and 90, that when the arms 53 and 54 are rocked downwardly by the cam 88 that the arm 90 will also be rocked downwardly, but if the arm 90 is obstructed at this time, the spring 91 will be compressed and therefore when the cam 88 clears the roll 89, the arms 53 and 54 will return upwardly until the pivot pin 68 is slightly above the plane passing through the shafts 56 and 69. The upward movement of the arms 53 and 54 at this time is of course limited by the pin 70 of the key detent of the special bank, but when the toggle composed of the arms 53 and 54 is in such position, in order to operate the machine it is necessary only to depress one of the special keys as was previously explained because the operation of the lever 1 brings the toggle to just the position to which it may be brought by spring 91.

In order to obstruct the movement of the arm 90 only when the lever 1 is in certain predetermined positions, the segment gear 24 which is fixed to the lever 1 meshes with a pinion 94 on a short shaft 95 which shaft also carries a disk 96 provided with a number of holes 97 corresponding to the number of totalizers and suitable for receiving pins 98 (Fig. 6). If the lever 1 is set to such position that one of the pins 98 is opposite the lower end 99 of the arm 90, then each time the machine is operated with the pin in this position, when the cam 88 engages the roller 89, the arm 90 will engage the pin causing the spring 91 to be compressed. Then when the cam 88 clears the roll 89, the spring 91 will force the arms 53 and 54 into such position that an operation of the lever 1 will not be required again to operate the machine, but the machine may be released by depression of any key of the special bank. The addition of this simple but effective mechanism tends to quickness of operation of the particular totalizers which are most used and as seen, the parts may be adjusted as desired so that any particular totalizer position will not need an operation of the setting lever.

The invention has been shown in connection with an electrically operated cash register, but clearly none of the electrical features are at all concerned, and the invention

is equally advantageous in other types of mechanism. The locking arm 37 has, as shown, functions of closing a circuit and permitting a clutching operation, but these functions are entirely aside and separable from its function of locking the machine, which latter is the only function it has in relation to the present invention. Clearly, whether the machine is driven by motor or by any connected mechanism or by hand, would require no change in the releasing mechanism of the machine, and it is therefore to be understood that the invention herein is not to be limited to any particular style of driving mechanism, except as definitely limited in certain of the claims.

From the foregoing description it will be evident that in the usual operation of the machine the main driving mechanism is normally locked by a means which is controlled by two separate manipulative devices, herein shown as a lever and any of a group of keys, and that therefore adjustment of two devices is necessary to permit release of the driving mechanism. By the provision of an adjustable device, control of the locking means by one manipulative device is destroyed and by the further provision of manually shiftable elements, namely, pins, the lever positions in which such control is destroyed may be regulated as desired.

In the operation of the improved machine, assuming the lever 1 adjusted so that a pin 98 is in position to cooperate with arm 90, the straightening of the toggle by cam 88 will tension spring 91 and after cam 88 passes roller 89, the spring 91 will expand and break the toggle, leaving the machine at the end of the operation, with lever 73 engaging pin 70 of the key detent, wherefore a mere depression of a special key will completely release the machine.

While the form of mechanism herein shown and described is admirably adapted to fulfil the objects primarily stated, it is to be understood that it is not intended to confine the invention to the one form of embodiment herein shown and described, as it is susceptible of embodiment in various forms all coming within the scope of the claims which follow.

What is claimed is as follows:—

1. In a machine of the class described, the combination with a main operating mechanism, means normally locking the same against operation, and two different manipulative devices each having connections controlling the release of said locking means; of adjustable means for destroying the control of one of said manipulative devices over said locking means.

2. In a machine of the class described, the combination with a main operating mechanism, means normally locking the same, and two different manipulative devices each hav-

ing connections controlling the release of said locking means; of an adjustable lever and connections therefrom for destroying the control of one of said manipulative devices over said locking means.

3. In a machine of the class described, the combination with a main operating mechanism, means normally locking the same, and two different manipulative devices each having connections controlling the release of said locking means, of a movable means having connections manually adjustable to destroy the control of one of said manipulative devices over said locking means in any desired position of said movable means.

4. In a machine of the class described, the combination with a main operating mechanism, means normally locking the same, and two different manipulative devices each having connections for controlling said locking means; of an adjustable lever and connections including devices manually shiftable to destroy the control of one of said manipulative devices over the locking means in any desired position of said lever.

5. In a machine of the class described, the combination with a main operating mechanism, means for normally locking same and two manipulative devices each having connections for controlling said locking means; of an adjustable lever and connections including a support and pins carried thereby and manually shiftable to destroy the control of one of said manipulative devices over the locking means in any desired position of said lever.

6. In a machine of the class described, the combination with a main operating mechanism and means including a toggle for locking said mechanism, of connections from said operating mechanism for straightening said toggle to locking position, and an adjustable device having connections to prevent said toggle from remaining in straightened position.

7. In a machine of the class described, the combination with a main operating mechanism and means for locking the same, of connections from said operating mechanism to force said locking means to locking position, and an adjustable device having connections which in certain positions of said device will engage an element of said locking means and move the same toward releasing position.

8. In a machine of the class described, the combination with a main operating mechanism and means for locking the same, of connections from said operating mechanism for forcing said locking means to locking position, an adjustable device having projections, and connections controlled by said projections for engaging an element of said locking means and moving the same toward releasing position.

9. In a machine of the class described, the combination with a main operating mechanism and means for locking the same, of connections from said operating mechanism
5 for forcing said locking means to locking position, an adjustable device having projections, an arm positioned to be engaged by said projections, and a spring compressed between said arm and an element of said
10 locking means.

10. In a machine of the class described, the combination with a main operating mechanism and means for locking the same, of connections from said operating mechanism for forcing said locking means to locking
15 position, an adjustable device having shiftable pins, an arm positioned to be engaged by said pins, and a spring compressed between said arm and an element of said
20 locking means.

11. In a machine of the class described, the combination with a main operating mechanism, and means including a toggle for locking the same, of connections from
25 said main operating mechanism for straightening said toggle, an adjustable device having projections, an arm positioned to be engaged by said projections, and a spring compressed between said arm and one of the toggle
30 arms.

12. In a machine of the class described, the combination with a main operating mechanism and means including a toggle for locking the same, of connections from said
35 main operating mechanism for straightening said toggle, a hand lever, and a device having projections and moved with said lever, an arm positioned to be engaged by said projections, and a spring compressed between
40 said arm and one of the toggle arms.

13. In a machine of the class described, the combination with a main operating mechanism, and means including a toggle for normally locking the same, of connections from said main operating mechanism
45 for straightening said toggle, a lever and devices mounted thereon for breaking said toggle, keys having connections controlling the movement of said locking means after
50 said toggle is broken, an adjustable device controlled by said lever and having projec-

tions, an arm positioned to engage said projections, and a spring between said arm and one of said toggle arms.

14. In a machine of the class described, 55 the combination with an accounting device, of a main operating mechanism, two adjustable setting members, a lock for the operating mechanism, connections between
60 said adjustable setting members and the operating mechanism normally requiring operations of both of the adjustable setting members to release the operating mechanism from the lock, and means for destroying the
65 control of one of the setting members over the locking mechanism.

15. In a machine of the class described, the combination with a main operating mechanism, of means for normally locking
70 same, two manipulative devices each having connections for controlling said locking means, an adjustable lever, one of said manipulative devices being carried by said lever, and connections including devices shiftable
75 to destroy the control of one of said manipulative devices over the locking means in any desired position of said lever.

16. In a machine of the class described, the combination with a main operating mechanism and means for locking the same, 80 of connections from said operating mechanism to force said locking means to locking position, an adjustable device, and a spring interposed between said adjustable device and said locking means to return the lock-
85 ing means out of locking position in certain positions of said adjustable device.

17. In a machine of the class described, the combination with an operating mechanism, of a lock for the operating mechanism, 90 a manipulative device for controlling said lock to move it to unlocking position, means controlled by the operating mechanism for moving the lock to locking position, and an adjustable device for making said latter
95 means ineffective.

In testimony whereof I affix my signature in the presence of two witnesses.

EDWARD J. VON PEIN.

Witnesses:

F. E. HAMILTON,
R. RUMMLER.