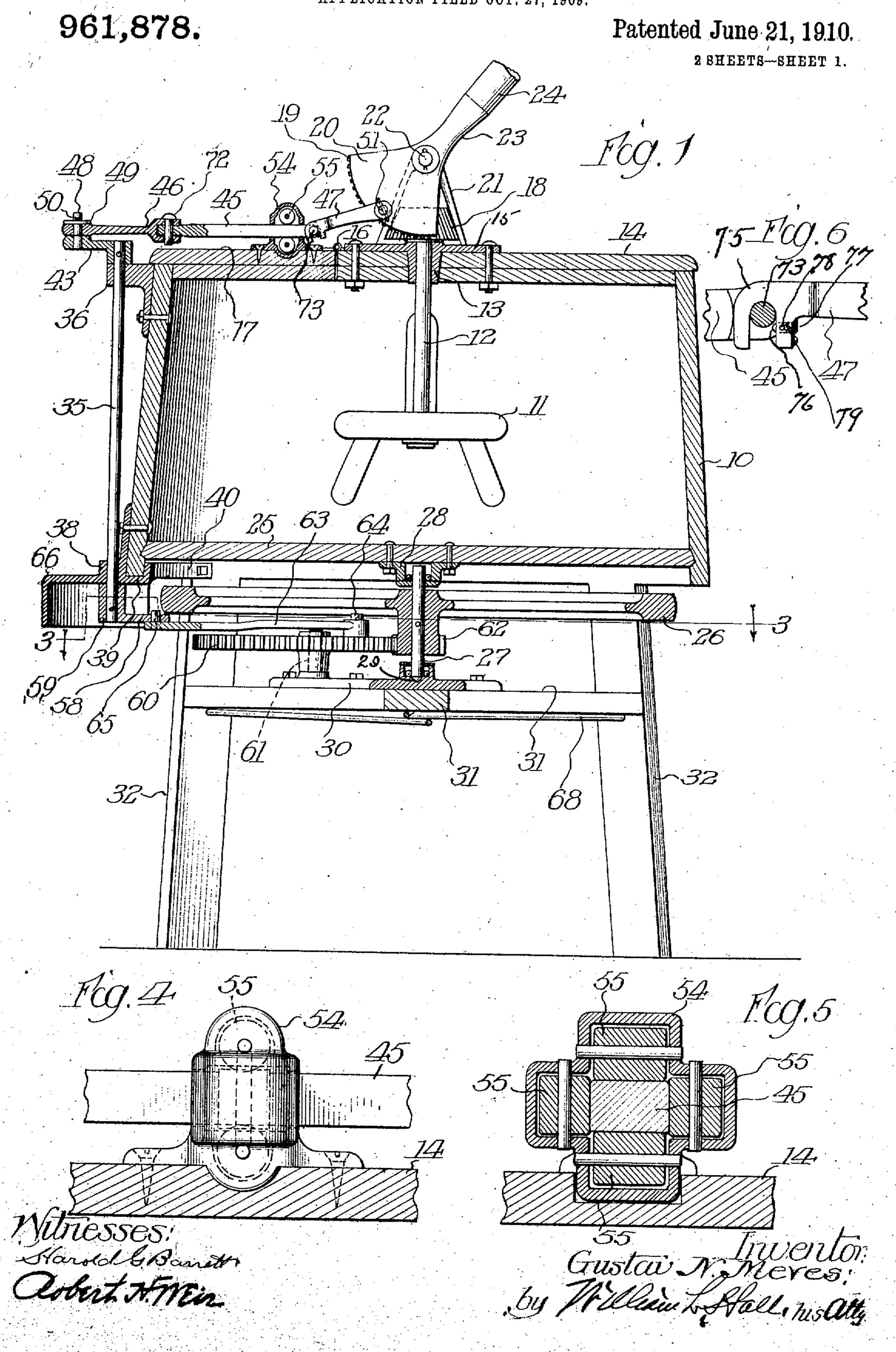
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OPERATING MECHANISM FOR WASHING MACHINES. APPLICATION FILED OCT. 27, 1909.



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UNITED STATES PATENT OFFICE.

GUSTAV N. MEVES, OF DAVENPORT, IOWA.

OPERATING MECHANISM FOR WASHING-MACHINES.

961,878.

Specification of Letters Patent. Patented June 21, 1910.

Application filed October 27, 1909. Serial No. 524,801.

To all whom it may concern:

Be it known that I, Gustav N. Meves, a citizen of the United States, and a resident of Davenport, in the county of Scott and 5 State of Iowa, have invented certain new and useful Improvements in Operating Mechanism for Washing-Machines; and I do hereby declare that the following is a full, clear, and exact description thereof, refer-10 ence being had to the accompanying drawings, and to the characters of reference marked thereon, which form a part of this specification.

This invention relates to improvements in 15 operating mechanism for washing machines of that type in which the stirrer head or dolly is given a rotary reciprocatory motion within the tub to turn the clothes first in one direction and then in the other through the

20 wash water in the tub.

The invention refers more specifically to improvements in machines of that type wherein a balance wheel or other power storing device is operatively connected with the 25 actuating device for the stirrer head shaft in such a way as to permit the mechanism to be driven with comparatively small power, when started in operation, and to relieve the mechanism of shocks and jars due to the 30 reversal of direction of the moving parts.

The invention is herein shown as applied. to a lever type of operating mechanism for washing machines, but may be applied to other types of such mechanisms, and may be 35 used in connection with operating mechanisms for other than washing machines.

As shown in the drawings, Figure 1 is a central vertical section of a washing machine equipped with an operating mechanism 40 embodying my invention. Fig. 2 is a bottom plan view thereof. Fig. 3 is a horizontal section on the line 3—3 of Fig. 1. Fig. 4 illustrates an anti-friction guide for the pitman associated with the stirrer shaft actu-45 ating device. Fig. 5 is a vertical sectional view of said guide. Fig. 6 is a detail illustrating the manner of detachably connecting the inner swinging member of the pitman with the reciprocatory member thereof.

As shown in the drawings, the tub 10, the stirrer 11 and the stirrer head shaft 12 are of the usual construction for this type of machine. The shaft 12 extends downwardly follows: 35 designates a vertical rotative through a bearing sleeve 13 that occupies an opening in the swinging lid or cover 14, and tub. It has rotative bearing at its upper is made integral with a plate 15 fixed to the lend in a bracket 36 that is bolted or other-

upper side of the swinging lid or cover. The swinging portion 14 of the top wall is hinged at 16 to the fixed or non-rising portion 17 thereof.

The stirrer head shaft is provided at its upper end with a beveled pinion 18 which meshes with the gear sector 19 of a segmental rocking member 20 that is pivoted to an upright bracket 21 which rises from 65 or is made integral with the plate 15, said bracket being provided with a horizontal pivot pin or stud 22 which extends through a bearing aperture in said rocking member. The said rocking member is provided with a 70 socket piece 23 which receives the lower end of a hand lever 24 by which the rocking member is swung or rocked about its axis and, through the gear sector and beveled pinion, imparts a rotary reciprocatory mo- 75 tion to the stirrer head shaft.

Located beneath the bottom 25 of the tub is a horizontal balance wheel 26. As shown, the said balance wheel is fixed to a vertical, rotative shaft 27 that is mounted 80 at its upper end in a bearing bracket 28. fixed to the lower side of the tub bottom, and at its lower end in a bearing cup 29 that is formed in a lower fixed plate 30. Said plate is shown as fixed to a support 8! arranged below the tub bottom, comprising cross members 31, 31 extending between and attached at their ends to the legs 32, 32 of the tub and intersecting each other in the central vertical axis of the balance 9 wheel. Suitable anti-friction media are interposed between the upper and lower ends of the shaft 27 and the bearing members to reduce the friction between said parts. Said balance wheel is operatively connect-9 ed with the rocking member 20 in such a way that the balance wheel has imparted thereto a continuous rotary motion in one direction, through the swinging or rocking movement of the segmental rocking mem-1 ber 20, whereby power is stored in the balance wheel to assist the actuation of the rocking member and stirrer head shaft, after the mechanism is started in operation, and to minimize the shocks brought on the parts at the points of reversal thereof.

The connection between the actuating member and the balance wheel is made as shaft which is located at one side of the

wise fixed to the side of the tub near the top thereof. At its lower end the shaft has bearing in a bracket 38 that is fixed at its central portion by a hook shaped flange 5 39 to the lower edges of the tub staves, beneath the bottom 25, and is provided with oppositely extending, curved arms 40 which extend to adjacent legs 32 of the tub and are attached thereto by bolts 41 or other

10 suitable fastening means.

To the upper end of the shaft 35 is fixed a crank 43. The crank 43 is connected to the rocking actuating member 20 through the medium of a jointed pitman compris-15 ing a central pitman member 45, and end pitman members 46, 47, which latter are pivotally connected to the ends of the central pitman member 45 and are pivotally connected at their outer ends to the crank 20 43 and the rocking actuating member 20, respectively. The pivotal connection between the outer pitman member and the crank 45 consists of a stud 48 which extends upwardly from the crank through an aper-25 ture in the outer pitman member, and the pitman member is held from rising thereon by means of a washer 49 surrounding the stud and a pin 50 extending through the stud above the washer. The pivotal connection 30 between the inner pitman member and the rocking actuating sector 20 comprises a stud 51 on the rocking actuating member that extends through a bearing aperture in said inner pitman member; the pitman member being 35 held on the stud in any suitable or convenient manner.

The central or intermediate pitman member 45 extends through and is guided by an anti-friction guide device comprising a cas-40 ing 54 attached to the upper side of the lid 17 in any suitable manner and is provided with a horizontal opening through which the pitman member 45 extends. Said casing is also provided with supports for anti-45 friction rollers 55, 55 which are arranged to bear on the lateral and top and bottom sides of said intermediate pitman member, thereby guiding the intermediate pitman member in a direct to and fro path, and 50 reducing the friction between said member and guide to a minimum.

The rotative shaft 35 is provided at its lower end with a crank 58, the hub or sleeve 59 of which is non-rotatively fixed to the 55 lower end of the shaft in any suitable manlower end of the shaft in any suitable

manner. 60 designates a large gear wheel ar-ranged horizontally beneath the balance 60 wheel and rotatively mounted on a stud 61 that extends upwardly from the frame plate or bearing member 30. The said gear wheel meshes with a pinion 62 that is carried by or formed integral with the lower end of 65 the hub of the balance wheel. The gear

wheel 60 is operatively connected with the lower crank 58 by means of a pitman 63 which is pivotally connected at its inner end with a pin or stud 64 mounted on and located eccentrically with respect to the gear 70 wheel, and is pivotally connected at its outer end with a stud 65 that is carried by the

said crank 58. A shield or hood 66, that may be made integral with the lower bearing bracket 38 for 75 the shaft 35, incloses the lower crank 58 to prevent the crank from striking a person standing near the machine during the outward throw or swing of the crank. The legs 32 of the tub may be held together 80 by tension rods 68, each extending between opposite legs and the rods crossing each other at the intersection of the cross members 31. The said rods extend at their ends through the legs and are screw-threaded to 85 receive nuts 69 by which the rods are placed under tension and the tub legs held firmly against the ends of the cross members to give the required rigidity to the connection between the cross members and legs. The 90 said cross members may be attached to the legs by screws 70 (Fig. 3) that extend through the legs and into the ends of the cross members.

In the operation of the machine, the hand 95 lever 24 is grasped by the operator and swung from side to side and operates, through the rocking gear sector member 20 and the pinion 18, to impart rotary reciprocatory motion to the stirrer head shaft. 100 During the rocking motion of the gear sector member, the jointed pitman is reciprocated endwise and operates, through the crank 43, to rotate the vertical shaft 35 continuously in one direction, and the ro- 105 tative motion of said shaft is transmitted, through the lower crank 58, the pitman 63, the gear 60 and pinion 62, to rotate the balance wheel at high speed in one direction. The momentum of the balance wheel, thus 110 rotated at high speed in one direction, serves to store power in the balance wheel, which is transmitted, through the connections described back to the hand actuated rocking gear sector member 20 and acts to materially 115 lessen the power required to actuate the rotary reciprocatory shaft after the mechanism has once been started in operation. Said arrangement also serves to continue the operation of the parts for a time after manual 120 power acting on the hand lever is discontinued. It will be noted that the outer member 46 of the jointed pitman is pivotally connected with the central part or member thereof, as by means of the pivot pin 72, 125 to swing horizontally and that the inner member 47 of said jointed pitman is pivotally connected with said central member, as by the horizontal pivot pin 73, to swing vertically. Thus the central member 45 of 13°

the jointed pitman reciprocates in a straight path, during the swinging movement of the actuating gear sector member, with a mininum binding stress of said central pitman 5 member against its anti-friction guide.

In order that the swinging portion 14 of the top wall or lid of the tub may be freely swung upwardly to permit access to the tub, provisions may be made to disconnect the 10 inner swinging member 47 of the jointed pitman from the central member 45 thereof. Such disconnection may be afforded by any suitable construction, such for instance as that illustrated in Fig. 6. As therein shown 15 the outer pivoted end of the inner crank member is formed with a hook 75 that engages over the pivot pin 73, and is provided with a spring-pressed latch bolt 76 that occupies a recess in said inner swinging member 20 and is normally forced, by a spring 77, arranged behind said recess and bearing against the end of the latch bolt, with its outer end beneath the pivot pin 73 so as to prevent the outer end of said inner member of the 25 jointed pitman from rising. The latch bolt is retracted, against the action of said spring 77, to free it from said pivot pin by means of a pin 78 extending outwardly therefrom through a slot 79 in the hook end of said 30 inner pivoted end of the pitman. Said pin also serves by contact with the outer end of the slot, to limit the outward movement of the latch bolt.

While I have described the structural de-35 tails of the mechanism shown with considerable particularity, it will be understood that, the details of the mechanism may be somewhat waried within the spirit and scope of the invention, and the invention is not lim-40 ited to the details shown, except as hereinafter made the subject of specific claims.

I claim as my invention:

1. Operating mechanism for washing and like machines comprising, in combination, a 45 shaft capable of rotary reciprocatory motion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel, and speed increasing gear mechanism operatively connected between said actuating 50 means and said balance wheel, embracing a shaft capable of rotating continuously in one direction and provided at both ends with cranks through which it is connected with said balance wheel and said actuating 55 device.

2. Operating mechanism for washing and like machines comprising, in combination, a vertical shaft capable of rotary reciprocatory motion, actuating means for imparting 60 reciprocatory motion to said shaft, a balance wheel arranged below said shaft and speed increasing gear mechanism operatively connected between said actuating means and said balance wheel, embracing a vertical shaft capable of rotating continuously in

one direction and provided at both ends with cranks through which said shaft is connected with said balance wheel and said actuating device.

3. Operating mechanism for washing and 70 like machines comprising, in combination, a vertical shaft capable of rotary reciprocatory motion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel arranged below said shaft and 75 speed increasing gear mechanism operatively connected between said actuating means and said balance wheel, embracing a vertical, rotative shaft provided at both ends with cranks and pivoted pitmen connecting said 80 cranks with said balance wheel and with

said actuating means. 4. Operating mechanism for washing and like machines comprising, in combination, a vertical shaft capable of rotary recipro- 85 catory motion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel arranged below the shaft and speed increasing gear mechanism operatively connected between said actuat- 90 ing means and said balance wheel, embracing a vertical, rotative shaft provided at its upper end with a crank, a pitman connecting said crank and actuating device, and speed increasing gears arranged between 95 and connecting the lower end of said rotative shaft with said balance wheel.

5. Operating mechanism for washing and like machines comprising, in combination, a vertical shaft capable of rotary recipro- 100 catory motion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel arranged below the shaft and speed increasing gear mechanism operatively connected between said actuat- 105 ing means and said balance wheel, embracing a vertical, rotative shaft operatively connected at its upper end with said actuating means, a crank at the lower end of said rotative shaft and a pitman connecting said 110 crank and balance wheel.

6. Operating mechanism for washing and like machines comprising, in combination, a shaft capable of rotary reciprocatory motion, actuating means for imparting rotary 115 reciprocatory motion to said shaft, a balance wheel, a rotative shaft provided at one end with a crank, a jointed pitman connecting said crank with the actuating means and speed increasing gear mechanism connected 120 between the other end of said shaft and the balance wheel.

7. Operating mechanism for washing and like machines comprising, in combination, a shaft capable of rotary reciprocatory mo- 125 tion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel, a rotative shaft, speed increasing gears connected between one end of said rotative shaft and said balance wheel, a crank 130

at the other end of said rotative shaft and a jointed pitman connecting said crank and said actuating means comprising a central portion which reciprocates in a straight path. 5 and end portions pivotally connected at their inner ends with the central portion and at their outer ends with said crank and

with said actuating means.

8. Operating mechanism for washing and 10 like machines comprising, in combination, a shaft capable of rotary reciprocatory motion, actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel, a rotative shaft, speed increasing 15 gears connected between one end of said rotative shaft and said balance wheel, a crank at the other end of said rotative shaft, a jointed pitman connecting said crank and said actuating means comprising a central 20 portion which reciprocates in a straight path and end portions pivotally connected at their inner ends with the central portion and at their outer ends with said crank and with said actuating means, and an anti-friction 25 guide bearing for the central portion of said

jointed pitman.

9. Operating mechanism for washing and like machines comprising, in combination, a shaft capable of rotary reciprocatory motion, 30 actuating means for imparting rotary reciprocatory motion to said shaft, a balance wheel, a rotative shaft, speed increasing gears connected between one end of said rotative shaft and said balance wheel, a crank 35 at the other end of said rotative shaft, a jointed pitman connecting said crank and said actuating means comprising a central portion which reciprocates in a straight path and end portions pivotally connected at their 40 inner ends with the central portion and at their outer ends with said crank and with said actuating means, and a guide bearing for the central portion of the jointed pitman comprising a casing or housing through 45 which said central portion of the pitman extends and bearing rollers mounted in said casing or housing and arranged to bear against the lateral and upper and lower sides of said central portion of the pitman.

50 10. Operating mechanism for washing and like machines comprising, in combination, a vertical shaft capable of rotary reciprocatory motion, actuating means for imparting

rotary reciprocatory motion to said shaft, a horizontally arranged balance wheel below said shaft, a vertical, rotative shaft operatively connected at its upper end with said actuating device, a crank at the lower end of said rotative shaft, said balance wheel being provided with a concentric pinion, a hori- 60 zontally arranged gear wheel meshing with said pinion and provided with an eccentrically located stud, and a pitman connecting

said crank with said stud.

11. Operating mechanism for washing and 65 like machines comprising, in combination, a shaft capable of rotary reciprocatory motion, a pinion fixed to said shaft, a rocking gear sector meshing with said pinion, a balance wheel provided with a concentric pin- 70 ion, a gear wheel meshing with the latter pinion and provided with an eccentrically arranged stud and operative connections between said rocking sector and said gear wheel comprising a rotative shaft provided 75 at both ends with cranks and links connecting said cranks with said stud and with said

rocking sector.

12. Operating mechanism for washing and like machines comprising, in combination, a 80 vertical shaft capable of rotary reciprocatory motion, a pinion fixed to said shaft, a rocking gear sector meshing with the pinion. a balance wheel arranged below said shaft, a vertical, rotative shaft operatively connected 85 at its lower end with said balance wheel, a horizontal crank fixed to the upper end of said rotative shaft and a jointed pitman connecting said rocking gear sector with the said crank comprising a central member 90 which reciprocates in a straight path, an outer member pivoted at one end to the outer end of said central member to swing horizontally and pivoted at its other end to said crank and an inner member pivoted at 95 one end to the inner end of said central member to swing vertically and pivoted at its other end to said rocking gear sector.

In testimony that I claim the foregoing as my invention I affix my signature in the 100 presence of two witnesses, this 23rd day of

October A. D. 1909.

GUSTAV N. MEVES.

Witnesses:

A. COLE BLANTON,

J. D. WALL.