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VALVE MECHANISM FOR DISCHARGING AIR AND WATER OF CONDENSATION FROM STEAM HEATING SYSTEMS.

951,751.

APPLICATION FILED MAR. 12, 1906.

Patented Mar. 8, 1910. 3 SHEETS-SHEET 1.

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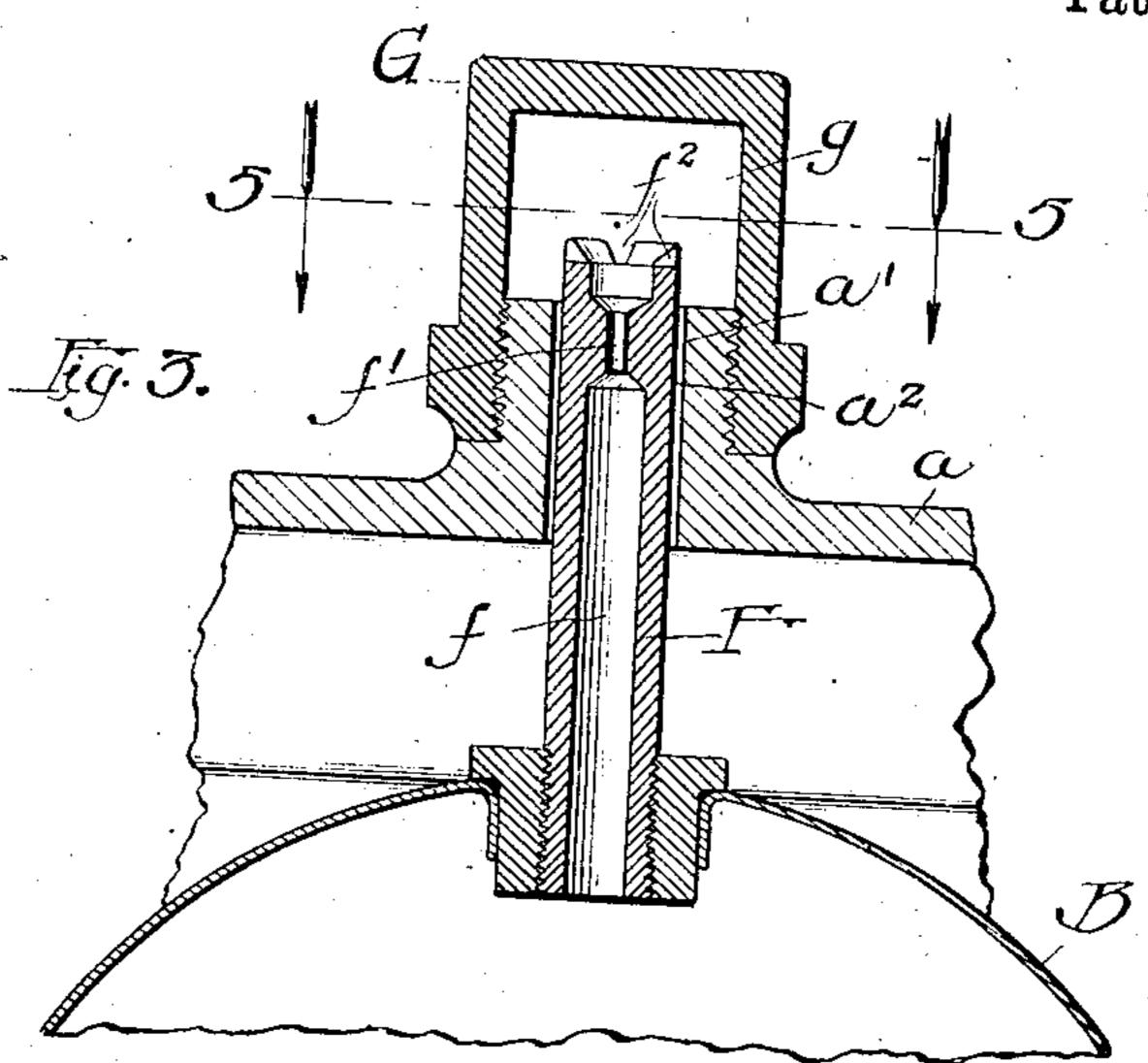
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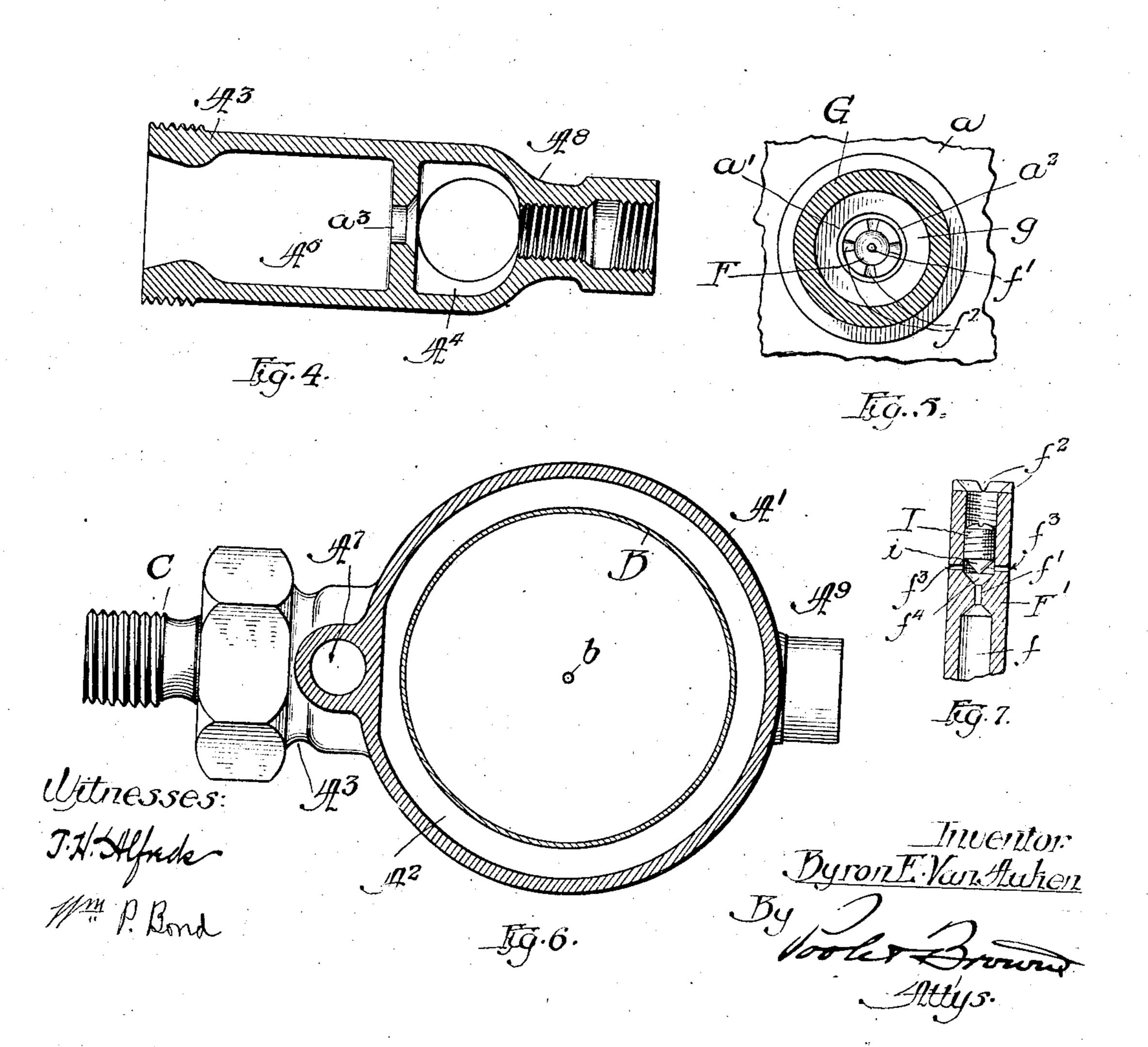
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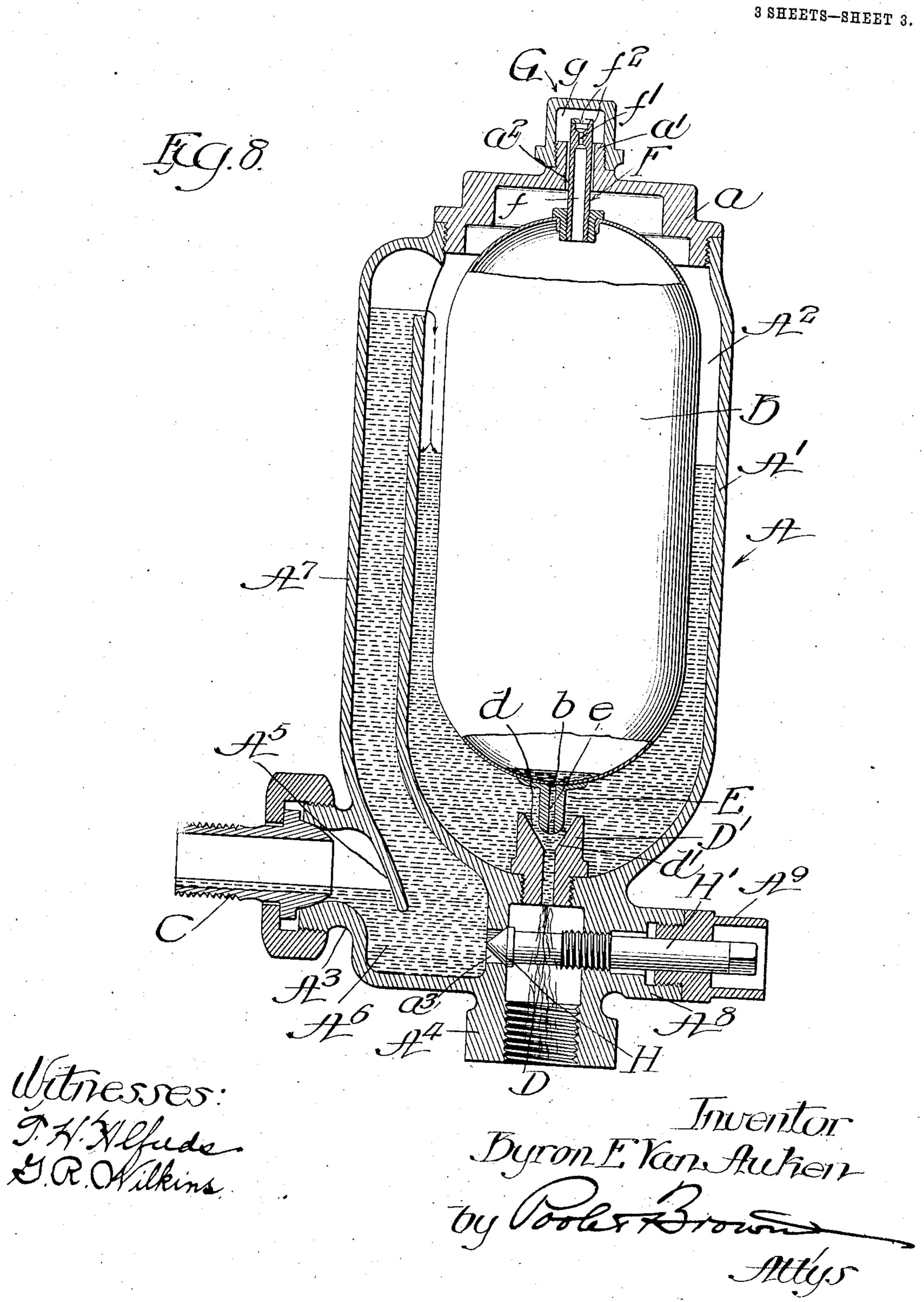
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UNITED STATES PATENT OFFICE.

BYRON E. VAN AUKEN, OF CHICAGO, ILLINOIS.

VALVE MECHANISM FOR DISCHARGING AIR AND WATER OF CONDENSATION FROM STEAM-HEATING SYSTEMS.

951,751.

Specification of Letters Patent.

Patented Mar. 8, 1910.

Application filed March 12, 1906. Serial No. 305,657.

To all whom it may concern:

Be it known that I, Byron E. Van Auken, a citizen of the United States, and a resident of the city of Chicago, in the county of Cook and State of Illinois, have invented certain new and useful Improvements in Valve Mechanism for Discharging Air and Water of Condensation from Steam-Heating Systems; and I do hereby declare that the following is a full, clear, and exact description thereof, reference being had to the accompanying drawings, and to the letters of reference marked thereon, which form a part of this specification.

This invention relates to improvements in valve mechanisms, in the nature of a float controlled steam trap, for discharging, by differential pressure, air and water of condensation from steam heating systems or from other inclosures or devices in which

steam condenses.

My improved valve mechanism, when applied to steam heating systems, is located between the return side of a heating unit or radiator of the system and the condensation water return pipe, which latter operates under a pressure less than that of the radiating or heating unit.

Among the objects of my invention is to produce a valve mechanism of this character which may be attached to a radiator of ordinary depth of leg or base, and so constructed as not to extend below the general level of the radiator base; to simplify and cheapen the cost of construction of such valve mechanism; to reduce the cost of maintenance thereof, and to improve and render certain its operation.

The invention consists in the matters herein set forth and more particularly point-

ed out in the appended claims.

As shown in the drawings,—Figure 1 is a vertical section of a valve mechanism made in accordance with my invention. Fig. 2 is a detail section, taken on line 2—2 of Fig. 1. Fig. 3 is an enlarged, fragmentary detail section of the upper part of the valve mechanism. Fig. 4 is a section, taken on the indirect line 4—4 of Fig. 1, with parts removed. Fig. 5 is a horizontal section, taken on line 5—5 of Fig. 3. Fig. 6 is a horizontal section, taken on line 6—6 of Fig. 1 Fig. 7 illustrates a modification of the means for discharging air. Fig. 8 is a vertical section of the valve mechanism, illustrating the de-

vice in the act of discharging water from the float chamber, the float being slightly raised.

In said drawings, A designates the valve mechanism, as a whole. It consists of a vertically elongated, chambered casting A1 in 60 which is formed a float chamber A². Said float chamber contains a vertically elongated, hollow float B, preferably made of thin sheet metal. The upper wall of the chamber comprises, as herein shown, a re- 65 movable cap or closure a, which permits the ready insertion and removal of the float. The casing or shell is provided near its bottom with a laterally directed inlet branch A³, and at its bottom with a vertically di- 70 rected, liquid discharge branch A*, the latter adapted for communication with a condensation water return pipe through which water of condensation is discharged from the float chamber. The said liquid dis- 75 charge pipe operates under a pressure considerably less than the normal pressure of the radiating units of the heating system. The said inset branch is adapted for connection with a radiator, or other condensing 80 device, through a short pipe C removably connected with the branch A3. The conduit formed by said pipe C and the branch A3 communicates with the float chamber through a seal at the inner end of the branch, 85 arranged to be sealed by the accumulation of water of condensation, and the air space in the upper part of the float chamber is adapted for open communication with the liquid discharge passage of the device, pref- 90 erably through a restricted opening, in a manner hereinafter to be described, thereby establishing and maintaining a differential pressure on opposite sides of the seal whereby a portion of the accumulated water is 95 intermittently discharged into said float chamber. The float chamber A is provided with a discharge passage D through which water, which collects in the float chamber, is intermittently discharged from the float 100 chamber to the discharge passage A* and thence to the condensation water pipe adapted for communication therewith. The said opening D is herein shown as formed in a vertical short tube or nipple D1 that extends 105 upwardly into the float chamber and is exteriorly screw-threaded at its lower end to engage a screw-thread opening in the lower wall of said float chamber. The discharge opening D of the float chamber is governed 110

by a valve E controlled by the float B. As herein shown, said valve consists of a short plug fixed rigidly to and depending from the lower end of the float B and formed at b its lower end to close the discharge passage D. The upper end of the tube or nipple D¹, in which is formed the discharge passage D, is provided with an upwardly and outwardly flaring recess d, and the lower end 10 of the valve E is correspondingly tapered to engage the flaring wall of said recess. The presence of said flaring or conical recess is advantageous, inasmuch as it guides the valve to its seat at the time the float is in-15 serted into the chamber and facilitates the insertion of the float in the chamber. The float is provided at its upper end with a guide-stem F which enters a suitable guide opening in a boss a^1 formed on the upper 20 wall of the removable cap or closure a of the float chamber, whereby the float is guided in its vertical movements.

In accordance with one of the principal features of the present invention, the discharge 25 passage of the device communicates with the air space in the upper part of the float chamber through suitable ports in the hollow float B. For this purpose, the stem F is shown as made hollow to provide a passage f which 30 communicates at its upper end with the float chamber and at its lower end with the interior of the hollow float; and the valve E is made hollow to provide a passage e which communicates at its upper end with the in-35 terior of the hollow float through an opening b in the bottom wall thereof and communicates at its lower end with the outlet passage D of the float chamber. The air passage through the float is restricted at one point. 40 As herein shown, the passage f is formed with a restricted portion or port f^1 which constitutes a contracted vein that limits the passage of air or air and vapor there-

through. It will be observed by reference to the drawings that the cross-sectional area of the said restricted opening or contracted vein f^1 is, relatively, very considerably less than the area of the discharge opening b whereby, so when the valve is seated and the interior of the float is in communication with the exhaust device through said opening b, there will exist a difference of pressure on opposite sides of the float wall. That is to say, 55 there will be less pressure in the interior of the float B than in the interior of the surrounding float chamber A². There may, for example, be atmospheric pressure in the float chamber and several inches of vacuum with-60 in the hollow float. It will thus be manifest that when the float B is raised, (see Fig. 8), and the opening b sealed by the temporary inrush of water of condensation, the air will continue to flow into the float through the re-85 stricted opening f^1 .

The seal in the conduit, through which water of condensation is discharged from the radiator into the float chamber, is shown as formed by a lip or weir A⁵ located just inside the inlet branch A³ and dipping into 70 a pocket A⁶ formed in the shell at the inner end of said branch, the lower edge of said weir being shown as located below the floor or bottom of the pipe C, or the radiator connected therewith. The location of the lower 75 edge of said lip or flange determines the water level in the pipe C and the base of the radiator adapted for communication therewith. Said seal may, however, be otherwise produced. In the present construction, the 80 pocket A⁶ communicates with the float chamber above the line of flotation of the float through a vertical passage A' located at one side of the float chamber, said passage discharging at its upper end into the float 85 chamber near the upper end of the latter. The said passage A⁷ and pocket A⁶ constitute, therefore, together with the pipe C and branch A³, a conduit designed to provide communication between the heating system 90 and the float chamber, and in which is located a liquid seal that is arranged to be sealed by the accumulation of the water of condensation in said conduit, whereby the superior pressure in the heating system 95 serves to force a portion of the water accumulating in said conduit into the float chamber and to discharge the air from said heating system with a minimum loss of steam.

The illustrated construction for estab- 100 lishing communication between the upper end of said hollow stem F and the upper part of the float chamber, is made as follows: The said stem extends loosely through an aperture in the boss a^1 into, and commu- 105 cates with, a chamber g formed in a hollow cap G that fits over and has screw-threaded engagement with the boss a^1 . The said chamber g communicates with the upper end of the float chamber through an annular pas- 110 sage a^2 formed between the loosely fitting stem F and the wall of the aperture through which it extends. The upper end of said stem is provided with notches f^2 adapted to be engaged with a screw driver or like im- 115 plement to turn the stem out of its screwthreaded engagement with the float. Said notches also prevent the air passage being entirely closed when the float is raised to such height to engage the end thereof with 120 the upper wall of the closing cap G.

Means are provided for discharging the water of condensation from the pocket A⁶ and vertical passage A7 when desired, consisting, as herein shown, of a port or open- 125 ing a³ affording communication between the pocket and the discharge branch A4, said port being normally closed by a conical valve H, the stem H¹ of which extends across the discharge passage of the device and laterally 130

outwardly through a branch As and has | screw-threaded engagement therewith. A stuffing box A⁹ prevents the leakage of vapor, air, and water around said stem. A leak-5 age passage is provided at the lower end of the float chamber to permit the escape of water therefrom at times when the valve is out of service. As herein shown, the leakage passage is provided at the outlet valve for the 10 chamber by forming a groove d1 (Fig. 2) in the conical face of the valve seat. Obviously, the forming of such groove in the valve É at this point will produce the same

result. In the operation of the valve mechanism, the upper part of the float chamber is in open communication with the discharge passage thereof through the hollow float at times when the valve governing the dis-20 charge of water from the float chamber is closed. The water of condensation from the heating system accumulates in the conduit leading to the float chamber and said conduit is sealed by the accumulation of said 25 water of condensation therein, thereby operating, in connection with the lower pressure of the discharge side of the valve mechanism, to increase the differential pressure on the opposite sides of said seal. The superior 30 pressure on the radiator side of the seal operates periodically to force the accumulated water of condensation into the float chamber until a sufficient quantity of water has been deposited in said chamber to raise the float 35 and the outlet valve E, whereupon water escapes from said float chamber through the discharge passage thereof until the body of water in said chamber is lowered below the line of flotation of the float. Thereupon the float is lowered by gravity and the valve E is seated to cut off the escape of water from the float chamber. During the period of accumulation of water in the float chamber, air and vapor finding their way into the 45 upper part of said chamber are conducted therefrom through the hollow float and the ports or passages thereof, and the differential pressures on the opposite sides of the seal, caused by the combined action of the 56 seal and the lower pressure into which the air and vapor is discharged, cause the water to be gradually forced into the float chamber until it accumulates therein in sufficient quantity to raise the float and valve 55 and permit the escape of a portion of the water from said float chamber, as described. In the normal operation of the form of the valve mechanism herein shown, wherein the water of condensation is directed into the top of the float chamber through the vertical passage A7, the water finds its level at the top of said passage, as shown in Fig. 1, and overflows into the float chamber, due to the superior pressure on the radiator side of the seal, assisted also by the disturbance

of the column of water in said passage A7. due to the momentary breaking of the seal and the passage of slugs of air and small quantities of steam beneath the weir A5 and upwardly through said water column. The 70 agitation of the water in said passage A7, caused by the rising of air and steam therethrough in the manner stated, has the effect of spilling water over the upper edge of the inner wall of said passage in greater or 75 less quantities, depending upon the violence of the disturbance of said water by the passage of the air and steam therethrough, as described. So long as the normal operation of the apparatus continues, therefore, the 80 passage $\tilde{\mathbf{A}}^{\bar{\tau}}$ will be substantially filled with water, as shown in Fig. 1. During the time the float is raised and the valve is open to permit the escape of water from the float chamber, the lower end of the hollow valve 85 is cut out of communication with the liquid discharge pipe, so that at this time the escape of air through the hollow float is cut off. When the valve is seated the escape of water from the float chamber is cut off. It 90 will thus be seen that the action of the mechanism is intermittent to alternately discharge air and water therefrom. That is to say, the air is continuously drawn from the float chamber into the interior of the hollow 95 float through the restricted opening f^1 but is intermittently discharged from the hollow float through the opening b, the latter being alternately sealed or closed (see Fig. 8) by water and opened to the discharge D, (see 100 Fig. 1) while the contracted passage or vein is always open.

By reason of the fact that the interior of the hollow float is subject to the pressure of the discharge passage of the valve mechan- 105 ism when the float is down (see Fig. 1), and the water discharge valve is closed, and is cut off from said passage when the float is raised (see Fig. 8), and the valve is open, the pressure in said float is subject to con- 110 siderable variation during a given cycle of the operation of the valve mechanism. That is to say, when the valve is closed, by reason of the restricted size of the opening f^1 the interior pressure of the float is less than that 115 of the float chamber, so that, when the float is raised and the passage of air cut off through the hollow valve E, the superior pressure in the float chamber outside the float forces the air into the space of lower 120 pressure within the float, thereby decreasing the differential pressure on the opposite sides of the wall of the float. When the float drops and the valve E again seats, the pressure will be again decreased to correspond 125 with the outlet side of the valve mechanism, thereby increasing the differential pressures on the opposite sides of the wall of the hollow float. It will thus be seen that air or air and vapor is constantly passing from the 130

float chamber to the interior of the float, and that it is intermittently discharged from the float at times when the valve E is seated.

The extension of the stem F into the chamber formed in the removable cap, as described, or other construction exposing the
end of the stem when the cap is removed, affords ready access to the passage or port
thereof to clear the same of any obstruction
that may lodge therein. Such passage may
be conveniently cleared by inserting therethrough a wire or like implement without
removal of the float or stem from place.

The provision of a passage at the lower 15 end of the hollow float is of considerable advantage, aside from forming part of the passage for the escape of air from the float chamber, inasmuch as this construction avoids the disadvantage of the leakage of 20 water into the hollow float, such as occurs in constructions wherein the hollow float is without such outlet. In the present construction, water leaking into the hollow float is drawn off as rapidly as it accumulates 25 therein, whereas in a construction wherein the float has no such opening, the leakage of water thereis to acts to vary the weight of the float, with the result of varying its action.

In Fig. 7 I have shown a construction in the air passage leading to the upper end of the hollow float whereby the capacity of the passage may be varied or regulated. In this construction, the stem F¹, corresponding to the stem F shown in the other figures, is provided with a screw-threaded plug I which enters the same and has screw-threaded engagement therewith. In this instance, the communication between the interior of the stem and the float chamber is provided through the medium of radial ports f^3 leading through the walls of the stem below the screw-threaded plug I. Said plug has a conical end i that opposes a conical surface 45 f in the stem F below said ports. By moving said plug upwardly or downwardly the capacity of the passage may be varied, as is obvious.

It will be noted that by reason of the re-50 stricted opening leading from the float chamber to the interior of the float and by reason of the larger opening leading from the interior of the float to the liquid discharge passage, water will more freely pass through the lower opening from the float chamber into the float when the float is raised than air or vapor will pass from the float chamber into the float through the upper restricted opening. The construction is such that the upper opening leading from the float chamber into the float is so restricted as compared to the lower opening that water will more freely pass through the lower opening than will the air or vapor through the upper

raised it is light but when the water flows into the lower end of the float the float becomes heavy. The float is thus rendered more certain of operation because the buoyancy of the water is assisted by the lightness of the float and the weight of the water within the lower end of the float assists in returning the float so as to close the discharge opening from the float chamber.

I claim as my invention:—

1. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a float chamber, a liquid discharge passage communicating therewith, a conduit 80 adapted to provide communication between a radiator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pres- 85 sure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a hollow float for governing said discharge passage, said float being provided 90 with a restricted air inlet opening and with a comparatively unrestricted outlet opening affording communication, through the hollow float, between the float chamber and a zone of lower pressure, whereby a partial 95 vacuum is established in the float when said liquid discharge passage is closed and whereby water is forced into the float when the latter is raised and said liquid discharge passage is in open communication with the float 100 chamber.

2. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a float chamber, a liquid discharge 105 passage communicating therewith, a hollow float having a valve for governing said liquid discharge passage, a conduit adapted to provide communication between a radiator and said float chamber, a liquid seal 110 arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of con- 115 densation is forced into said float chamber, the said float being provided with a restricted air inlet opening and with a comparatively unrestricted outlet opening affording communication, through the hollow float, be- 120 tween the float chamber and a zone of lower pressure whereby a partial vacuum is established in the float when said liquid discharge passage is closed and whereby water is forced into the float when the latter is 125 raised and said liquid discharge passage is in open communication with the float chamber.

than will the air or vapor through the upper 3. Valve mechanism for discharging air opening. It follows that when the float is and water of condensation from steam heat 180

ing systems by differential pressure comprising a float chamber, a liquid discharge passage communicating therewith, a conduit adapted to provide communication between 5 a radiator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said seal whereby 10 a portion of the accumulated water of condensation is forced into said float chamber, a hollow float in said float chamber provided with a valve for controlling said liquid discharge passage, said float being provided 15 with an air passage extending through the float and valve and discharging into said liquid discharge passage, the upper part of said air passage being a restricted inlet opening and the lower part being a compara-20 tively unrestricted outlet opening through the valve whereby a partial vacuum is established in the float when the valve is seated and whereby water is forced into the float from the float chamber when the valve is raised from its seat.

4. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a shell formed with a float chamber, 30 a liquid discharge passage communicating with said float chamber, a hollow float in said chamber, a valve carried by the float which governs the liquid discharge passage, a conduit adapted to provide communication be-35 tween a radiator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid 40 seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a hollow guide stem extending upwardly from said float and having guiding engagement with said shell and providing a restricted opening for the passage of air from the float chamber to the hollow float, said float being provided also with a comparatively unrestricted outlet opening affording communication between the hol-10 low float and a zone of lower pressure than that of the float chamber whereby a partial vacuum is established in the float when the valve carried thereby is seated and water is forced into the float from the float cham-5 ber when said valve is raised.

5. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a shell, a float chamber therein, a o conduit adapted to provide communication between a radiator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differen-5 tial pressure on opposite sides of said liquid |

seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a liquid discharge passage communicating with the float chamber, a hollow float adapted to govern said dis- 70 charge passage and provided with an air passage extending through the float and discharging into said liquid discharge passage, the upper part of said air discharge passage being a restricted inlet opening and 75 the lower part being a comparatively unrestricted outlet opening affording communication between the hollow float and a zone of lower pressure than that of the float chamber, whereby a partial vacuum is 80 established in the float when the liquid discharge passage is closed against communication with the float chamber and whereby water is forced into the float from the float chamber when the float is raised, and means 85 for partially closing the upper part of the air discharge passage, whereby the flow of air from the float chamber into the hollow float becomes more restricted when the float is raised.

6. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a float chamber, a conduit adapted to provide communication between a radi- 95 ator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby 100 a portion of the accumulated water of condensation is forced into said float chamber, a liquid discharge passage communicating with the float chamber, a hollow float, a valve carried by said float for governing 105 said discharge passage, said float being provided at its upper end with a restricted opening through which air is discharged from the float chamber to the interior hollow float and being provided at its lower 110 end with an opening through which air is discharged from the hollow float, through said valve, to the liquid discharge opening the lower opening of the float being sealed by the water in the float chamber when the 115 float is raised.

7. A valve mechanism for the purpose described, comprising a float chamber provided with liquid inlet and discharge passages, the latter operating under a pressure 120 less than that of the former, a hollow float in said chamber, a valve closure attached to said float for controlling the liquid discharge passage, said float being provided at its upper end with a restricted opening 125 through which air is discharged from the float chamber to the interior of the hollow float and being provided at its lower end with an opening through which air is discharged from the hollow float to the liquid 130

discharge opening, the lower opening of the float being sealed by the water in the float chamber when the float is raised.

8. Valve mechanism for discharging air 5 and water of condensation from steam heating systems by differential pressure, comprising a float chamber, a liquid discharge passage communicating therewith, a hollow float for governing said discharge passage, 10 a conduit adapted to provide communication between a radiator and said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit, thereby increasing the differential 15 pressure on opposite sides of said liquid seal, whereby a portion of the accumulated water of condensation is forced into said float chamber, said float being provided with a continuous air passage extending as an en-20 tirety through the float and discharging into said liquid discharge passage, the upper part of said passage being a restricted opening, and the lower part of said passage being adapted to be sealed by water in the float 25 chamber when the float is raised to release the water from said float chamber to said

liquid discharge passage. 9. Valve mechanism for discharging air and water of condensation from steam heat-30 ing systems by differential pressure comprising a shell, a float chamber therein, a conduit adapted to provide communication between a radiator and said float chamber, a liquid seal arranged to be sealed by the ac-35 cumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float 40 chamber, a liquid discharge passage communicating with said float chamber, a hollow float provided with a valve for con-

45 and provided with a restricted opening for the passage of air from the float chamber to the hollow float, said stem having guiding engagement with the top wall of the float chamber, a cap having detachable engage-50 ment with said top wall and inclosing the

trolling said liquid discharge passage, a

stem extending upwardly from said float

upper end of said stem, said float being provided with a comparatively unrestricted outlet opening through the valve carried by the float to afford communication between the 55 hollow float and a zone of lower pressure

than that of the float chamber, whereby a partial vacuum is established in the float when the valve is seated and water is forced into the float from the float chamber when 60 the valve is raised from its seat.

10. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure, comprising a float chamber, a conduit adapted \$5 to provide communication between a radi-

ator and said float chamber, a liquid discharge passage communicating with said float chamber, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit, thereby increas- 70 ing the differential pressure on opposite sides of said seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a hollow float in said float chamber provided with a valve 75 closure for controlling said discharge passage and provided also with an upper restricted air and vapor opening and a larger lower opening whereby water will more freely pass through the lower opening from 80 the float chamber into the float when the float is raised than the air or vapor will pass from the float chamber into the float through the said restricted opening.

11. Valve mechanism for discharging air 85 and water of condensation from steam heating systems by differential pressure comprising a float chamber, a conduit adapted to provide communication between a radiator and said float chamber, a liquid seal ar- 90 ranged to be sealed by the accumulation of water of condensation in the conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensa- 95 tion is forced into said float chamber, a liquid dischargé passage communicating with the float chamber, a hollow float for governing said discharge passage, said float being provided with an air passage whereby 100 air may be forced continuously from the float chamber to the interior of the float and being also provided with an air passage whereby air may be intermittently discharged from the interior of the float into 105 the liquid discharge passage, the air inlet passage being a restricted opening and the air outlet passage being a comparatively unrestricted opening.

12. Valve mechanism for discharging air 110 and water of condensation from steam heating systems by differential pressure comprising a float chamber, a liquid discharge passage communicating therewith, a conduit adapted to provide communication between 115 a radiator and said float chamber and opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increas- 120 ing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a hollow float for governing said discharge passage, said float 125 being provided with a restricted air inlet opening and with a comparatively unrestricted outlet opening affording communication, through the hollow float, between the float chamber and a zone of lower pressure 130

whereby a partial vacuum is established in the float when said liquid discharge passage is closed and whereby water is forced into the float when the latter is raised and said bliquid discharge passage is in open communication with the float all liquid discharge passage is in open commu-

nication with the float chamber.

13. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure compris-10 ing a float chamber, a liquid discharge passage communicating therewith, a conduit adapted to provide communication between a radiator and said float chamber and opening into said float chamber above the line of 15 flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the 20 accumulated water of condensation is forced into said float chamber, a hollow float chamber having a valve adapted to govern the liquid discharge passage and being provided with a restricted air inlet opening and with 25 a comparatively unrestricted outlet opening affording communication, through the hollow float, between the float chamber and a zone of lower pressure, whereby a partial vacuum is established in the float when said 30 liquid discharge passage is closed and whereby water is forced into the float when the latter is raised and said liquid discharge passage is in open communication with the float chamber.

14. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a float chamber, a liquid discharge passage communicating therewith, a conduit 40 adapted to provide communication between a radiator and said float chamber and opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of 45 water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a hollow 50 float provided with a valve for controlling said liquid discharge passage and being also provided with an air passage extending through the float and valve and discharging into said liquid discharge passage, the is upper part of said air passage being a restricted inlet opening and the lower part being a comparatively unrestricted outlet opening through the valve, whereby a partial vacuum is established in the float when o the valve is seated and whereby water is | forced into the float from the float chamber when the valve is raised from its seat.

15. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure com-

prising a shell formed with a float chamber, a liquid discharge passage communicating with said float chamber, a hollow float in said chamber, a valve carried by the float which governs the liquid discharge passage, 70 a conduit adapted to provide communication between a radiator and said float chamber and opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumula- 75 tion of water of condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float 80 chamber, a hollow guide stem extending upwardly from said float and having guiding engagement with said shell and providing a restricted opening for the passage of air from the float chamber to the hollow float, 85 said float being provided also with a comparatively unrestricted outlet opening affording communication between the hollow float and a zone of lower pressure than that of the float chamber, whereby a partial vacuum 90 is established in the float when the valve carried thereby is seated and whereby water is forced into the float from the float cham-

ber when said valve is raised. 16. Valve mechanism for discharging air 95 and water of condensation from steam heating systems by differential pressure comprising a shell, a float chamber therein, a conduit adapted to provide communication between a radiator and said float chamber 100 and opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pres- 105 sure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a liquid discharge passage communicating with the float chamber, a hollow 110 float for governing the liquid discharge passage and provided with an air passage extending through the float and discharging into said liquid passage, the upper part of said air discharge passage being a restricted 115 inlet opening and the lower part being a comparatively unrestricted outlet opening affording communication between the hollow float and a zone of lower pressure than that of the float chamber, whereby a partial 120 vacuum is established in the float when the liquid discharge passage is closed against communication with the float chamber and whereby water is forced into the float from the float chamber when the float is raised, 125 and means for partially closing the upper part of the air discharge passage whereby the flow of air from the float chamber into the hollow float becomes more restricted when the float is raised.

17. Valve mechanism for discharging air and water of condensation from steam heating systems by differential pressure comprising a float chamber, a conduit adapted to 5 provide communication between a radiator and said float chamber and opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of 10 condensation in said conduit thereby increasing the differential pressure on opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a 15 liquid discharge passage communicating with the float chamber, a hollow float in said float chamber, a valve carried by said float for governing said discharge passage, said float being provided at its upper end with 20 a restricted opening and through which air is discharged from the float chamber to the interior of the hollow float and being provided at its lower end with an opening through which air is discharged from the 25 hollow float through said valve to the liquid discharge opening, the lower opening of the float being sealed by the water in the float chamber when the float is raised.

18. Valve mechanism for discharging air 30 and water of condensation from steam heating systems by differential pressure comprising a shell, a float chamber therein a conduit adapted to provide communication between a radiator and said float chamber and 35 opening into said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of condensation in said conduit thereby increasing the differential pressure on 40 opposite sides of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a liquid discharge passage communicating with said float chamber, a hollow float in 45 said float chamber provided with a valve for controlling said liquid discharge passage, a stem extending upwardly from said float and providing a restricted opening for the passage of air from the float chamber to the 50 hollow float, said stem having guiding engagement with the top wall of the float

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chamber, a cap having detachable engagement with said top wall and inclosing the upper end of said stem, said float being provided with a comparatively unrestricted out- 55 let opening through the valve carried by the float to afford communication between the hollow float and a zone of lower pressure than that of the float chamber, whereby a partial vacuum is established in the hollow 60 float when the valve is seated and whereby water is forced into the float from the float chamber when the valve is raised from its seat.

19. Valve mechanism for discharging air 65 and water of condensation from steam heating systems by differential pressure comprising a float chamber, a conduit adapted to provide communication between a radiator and said float chamber and opening into 70 said float chamber above the line of flotation of said float, a liquid seal arranged to be sealed by the accumulation of water of condensation in the conduit thereby increasing the differential pressure on opposite sides 75 of said liquid seal whereby a portion of the accumulated water of condensation is forced into said float chamber, a liquid discharge passage communicating with the float chamber, a hollow float for governing said dis- 80 charge passage, said float being provided with an air passage whereby air may be forced continuously from the float chamber into the interior of the float and being provided also with an air discharge passage whereby air 85 may be intermittently discharged from the interior of the float into the liquid discharge passage, the air inlet passage being a restricted opening and the air outlet passage being a comparatively unrestricted 90 opening, the float being adjusted whereby the said outlet opening of the float is sealed by the water in the float chamber when the float is raised.

In testimony, that I claim the foregoing 95 as my invention I affix my signature in presence of two witnesses, this 9th day of March A. D. 1906.

BYRON E. VAN AUKEN.

Witnesses:

TAYLOR E. BROWN, GERTRUDE J. BRYCE.