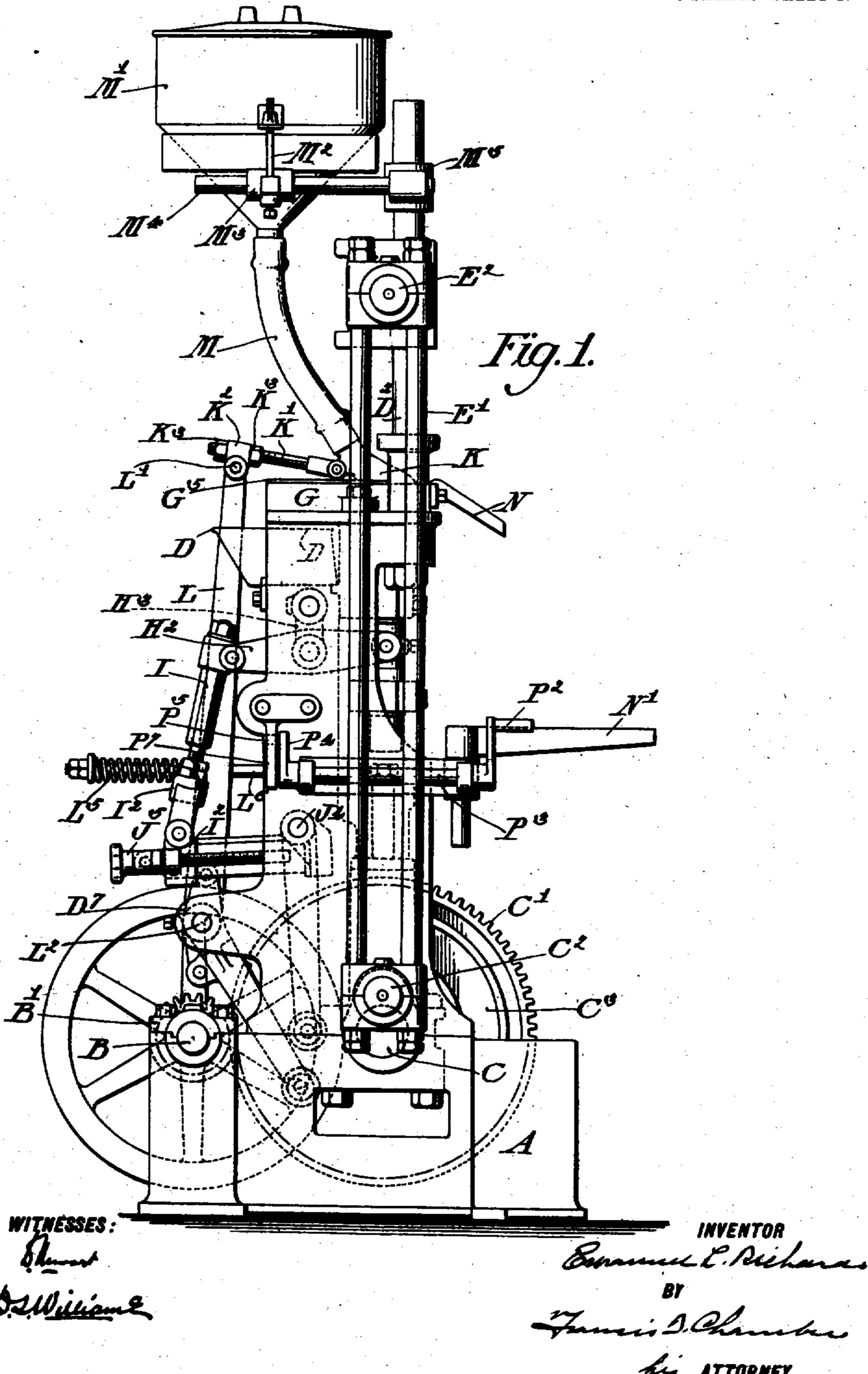
## E. L. RICHARDS. TABLET MACHINE. APPLICATION FILED APR. 23, 1907.

900,957.

Patented Oct. 13, 1908.

6 SHEETS-SHEET 1.



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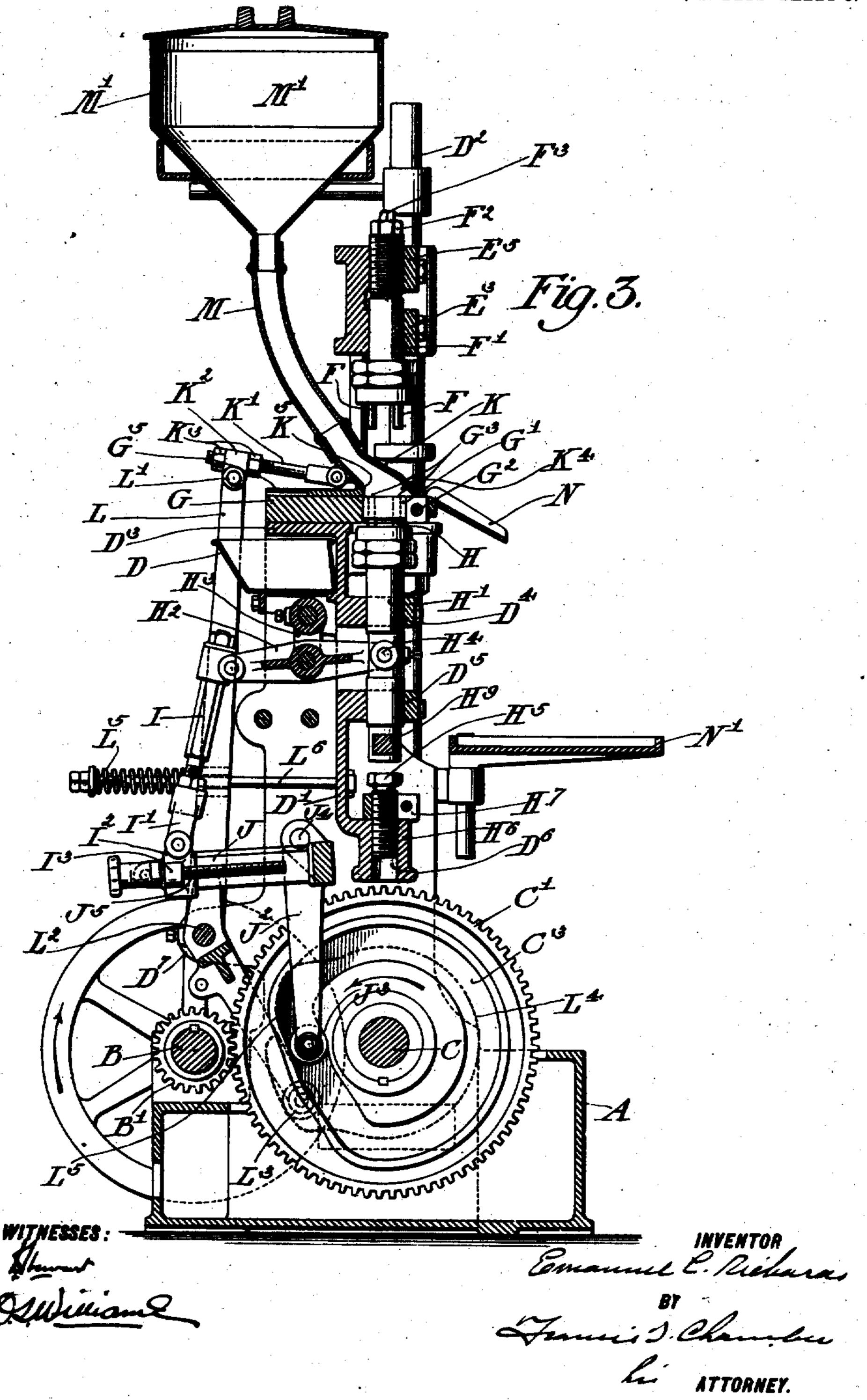
THE NORRIS PETERS CO., WASHINGTON, D.

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5 SHEETS-SHEET 3.



THE HORRIS PETERS CO., WASHINGTON, !

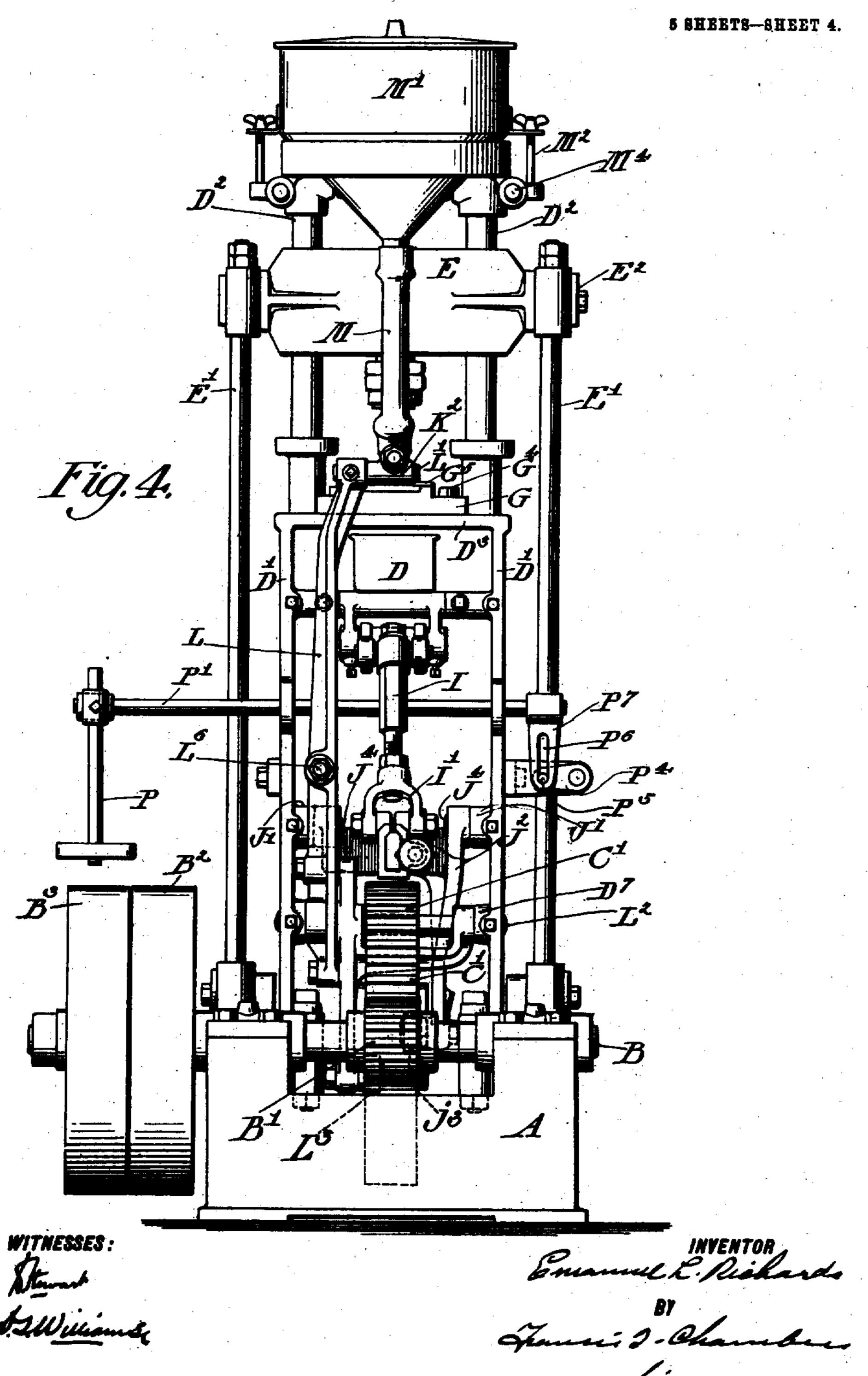
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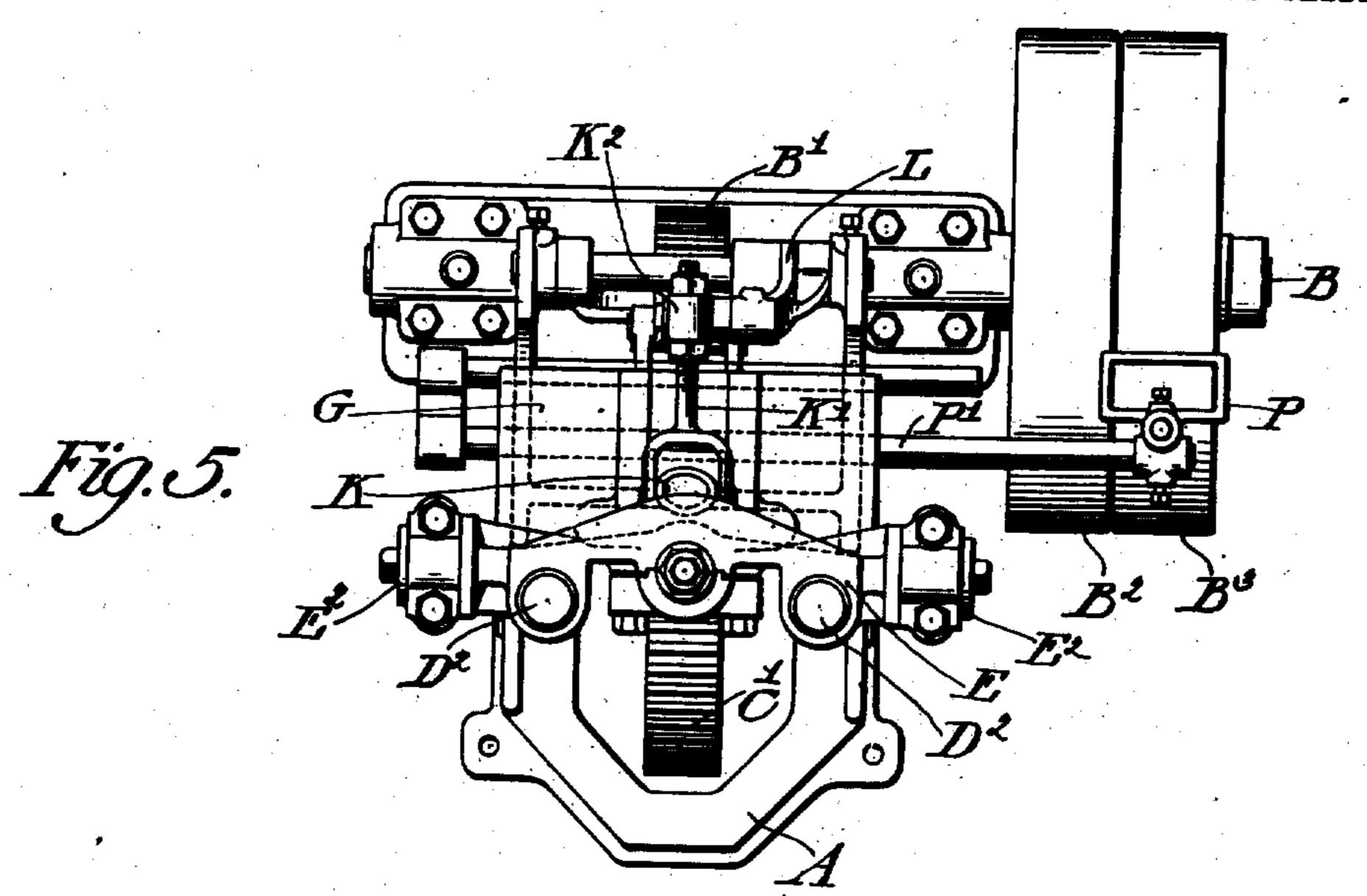
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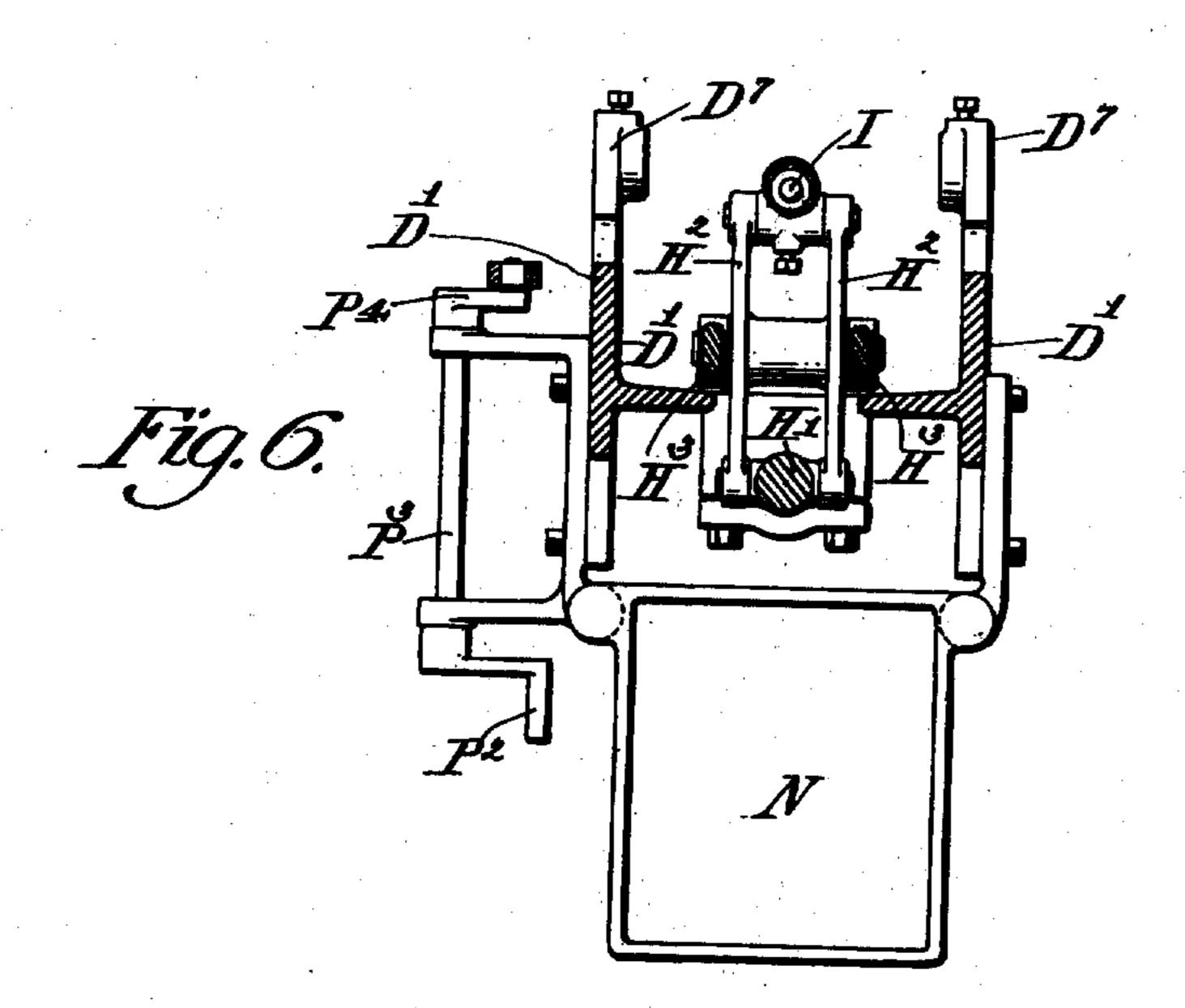
## E. L. RICHARDS. TABLET MACHINE. APPLICATION FILED APB. 23, 1907.

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5 SHEETS-SHEET 5.





WITNESSES: Described Emanuel E. Richard.

BY

Granici S. Runder.

Li ATTORNEY.

## UNITED STATES PATENT OFFICE.

EMANUEL L. RICHARDS, OF PHILADELPHIA, PENNSYLVANIA, ASSIGNOR TO JOHN WYETH AND BROTHER, INCORPORATED, OF PHILADELPHIA, PENNSYLVANIA, A CORPORA-TION OF PENNSYLVANIA.

TABLET-MACHINE.

No. 900,957.

Specification of Letters Patent.

Patented Oct. 13, 1908.

Application filed April 23, 1907. Serial No. 369,743.

To all whom it may concern:

Be it known that I, EMANUEL L. RICHin the city and county of Philadelphia, in 5 the State of Pennsylvania, have invented a certain new and useful Improvement in Tablet-Machines, of which the following is a true and exact description, reference being had to the accompanying drawings, which 10 form a part thereof.

My present invention relates to means for compressing medicinal tablets and the like from granular or powdered material.

The object of my invention is the con-15 struction of a machine for the purpose mentioned which will produce tablets of uniform quality and with little waste, which may be readily adjusted to give any desired degree of compression and tablets of any desired 20 thickness, and which will be simple, reliable and effective in its mechanical construction.

The various features of novelty which characterize my invention are pointed out with particularity in the claims annexed to 25 and forming a part of this specification. For a better understanding of my invention, however, and the advantages possessed by it, reference may be had to the accompanying drawings and descriptive matter, in which 30 I have illustrated and described a machine embodying my invention.

Of the drawings, Figure 1 is a side elevation of the machine. Fig. 2 is a front elevation. Fig. 3 is a section on the line 3-3 of 35 Fig. 2. Fig. 4 is a rear elevation. Fig. 5 is a plan view with some parts removed, and Fig. 6 is a section on the line 6—6 of Fig. 2.

In the drawings, A represents a base having formed in it the lower halves of the bear-49 ings for the driving shaft B and for the main operating shaft C, which is driven from the shaft B through gear wheels B<sup>1</sup> and C<sup>1</sup>. On the base A is bolted a main frame D comprising a pair of side plates D<sup>1</sup> and connect-45 ing portions. The upper halves of the bearings for the shaft C are formed in the frame D. A pair of parallel columns or guide posts D<sup>2</sup> extend from the upper end of the main frame D to which they are rigidly 50 connected. A crosshead E, mounted on the guide rods D<sup>2</sup> is reciprocated by means of connecting rods E<sup>1</sup> which are journaled at their upper end on trunnions E<sup>2</sup> carried on the crosshead and at their lower end on wrist 55 pins C<sup>2</sup> carried by, but eccentrically disposed with respect to the shaft C. The upper tab- | of the web D2 in line with the holder H6,

let forming plungers are in the form of pins F carried by a holder F, the shank of which ards, a citizen of the United States, residing [ is clamped in the crosshead E by means of a clamping yoke E<sup>3</sup> secured by bolts E<sup>4</sup>. The 60 upper end of the holder F' bears against and is supported by the lower end of a hollow bolt F<sup>2</sup> which is threaded in a nut formed partly by the crosshead E and partly by a clamping yoke E<sup>5</sup> adjustably secured to the 65 crosshead by bolts E<sup>\*</sup>. The position of the upper plungers relative to the crosshead may be varied by loosening the bolts E<sup>4</sup> and E<sup>6</sup> and adjusting the holder F<sup>1</sup> and the bolt F<sup>2</sup>. By tightening the bolts E<sup>4</sup> and E<sup>6</sup> the parts 70 are prevented from working loose. The bolt F<sup>2</sup> forms a means for obtaining an adjustment of the plungers more delicate, accurate and secure than would be otherwise possible. A bolt  $F^3$  passing through the hollow bolt  $F^2$  75 is threaded into a socket formed in the upper end of the holder F<sup>1</sup>. This prevents the holder F<sup>1</sup> from dropping and possibly injuring the plungers F when the clamping yokes are loosened. When it is desired to 80 remove the holder F<sup>1</sup> and then return it without disturbing the adjustment, as in cleaning, this may be readily done by adjustment of the bolt F<sup>3</sup> and clamping yoke E<sup>3</sup>.

> On the top cross D<sup>3</sup> of the frame D be- 85 tween the posts D<sup>2</sup> is secured a table G by bolts G4. The table G has an opening in line with the plungers F in which is secured a die block G<sup>1</sup>. Preferably the table G has a kerf at one side of the opening receiving the 90 die block G<sup>1</sup> to facilitate the clamping of the die in place by a clamping bolt G<sup>2</sup>. The die block G<sup>1</sup> is provided with passages G<sup>3</sup> into the upper ends of which the lower ends of the plungers F are adapted to enter, and 95 the tablets are formed in the passages G<sup>\*</sup> between the lower ends of the plungers F and the upper ends of similar plungers If which are carried by a holder H<sup>1</sup> longitudinally movable in line with the holder F<sup>1</sup> in 100 bearings formed in cross pieces D<sup>4</sup> and D<sup>5</sup> of the framework D. Levers H<sup>2</sup> supported from the framework D by links  $\bar{\mathbf{H}}^{\bar{s}}$  have their inner ends journaled on trunnions H<sup>4</sup> carried by the holder H<sup>1</sup>, and the oscillation 105 of the levers H<sup>2</sup> reciprocates the holder H<sup>1</sup> to move the plungers H up and down in the passages G<sup>3</sup>. The downward movement of the holder H<sup>1</sup> is definitely limited by an abutment in the form of a bolt H<sup>5</sup> projecting 110 from a socket D<sup>6</sup> formed in the framework

and secured in any desired position by a lock! nut H<sup>7</sup>. Similarly the upward movement of the holder H<sup>1</sup> is definitely limited by adjustable bolts H<sup>8</sup> carried in arms H<sup>9</sup> project-5 ing laterally from the lower end of the holder H1 and adapted to engage the trans-

verse frame rib D<sup>5</sup>.

The outer ends of the levers H<sup>2</sup> are connected by a link comprising relatively ad-10 justable parts I and I1 with a block or crosshead I<sup>2</sup> which is secured on one arm J of a bell crank lever pivoted on pins J' supported by bosses  $J^1$  of the frame work D. The other arm J<sup>2</sup> of the lever carries a cam roll 15 J<sup>3</sup> which travels in the cam groove C<sup>3</sup> formed in one side of the gear wheel C1. To vary the stroke of the plunger holder III, the block  ${f I}^2$  is made adjustable along the arm Jby means of a screw J<sup>5</sup> swiveled in the outer 20 end of the arm and passing through a threaded socket formed in the lug I3 of the block I<sup>2</sup>.

It will be observed that the center of the pivotal connection between the link member 25 I<sup>1</sup> and the block I<sup>2</sup> is so located that when the block I<sup>2</sup> is moved to the inner limit of its movement along the lever arm J, the pivotal center of the link connection is coaxial with the center of movement of the bell 30 crank lever about the stud J<sup>4</sup>. In consequence, the oscillation of the bell crank lever arm J<sup>2</sup> can be made to produce any desired amount of movement of the holder H<sup>1</sup> from

zero up to a maximum.

On the upper side of the table G are pro-- vided guideways G<sup>5</sup> in which reciprocates the feed shoe K. The shoe K has pivotally connected to it a rod or link K<sup>1</sup> on the outer end of which is adjustably secured a block 40 K² by means of nuts K³. One end of a lever L is pivotally connected to the block K<sup>2</sup> by the pin L<sup>1</sup>. The lever L is pivotally supported between its ends by a stud or shaft L<sup>2</sup> journaled in ears D' of the framework. The 45 lower end of the lever carries a cam roll L<sup>a</sup> which works in a cam groove L<sup>4</sup> formed in the opposite side of the gear wheel C<sup>t</sup> from that in which the cam groove  $C^3$  is formed. The cam roll L<sup>a</sup> is at all times held against 50 the cam L4 by a spring L5 held against the upper arm of the lever L by a bolt L secured to the framework of the web D<sup>1</sup>.

The shoe K has its upper end connected by a flexible tube M to the discharge end of 55 a hopper or receptacle M<sup>1</sup> secured by bolts M<sup>2</sup> to collars M<sup>3</sup> adjustable along transverse rods M<sup>4</sup> supported by collars M<sup>5</sup> adjustably secured in place on the upper end of the posts D. By this arrangement the hopper 60 or receptacle M1 may be adjusted toward and away from the level of the table G and also toward and away from the posts D. From the front end of the table G a chute N leads for conveying material discharged 65 from the front end of the table onto the

table N<sup>1</sup> secured to the framework B or into a receptacle not shown, supported on the table N<sup>1</sup>. A waste receptacle D is suitably supported by the framework D in position to receive material which may work out of the 70 feed shoe and be moved off of the rear edge of the table. A belt shipper P is supported by a shaft P<sup>1</sup> slidingly mounted in the side frames D and operated at the front of the machine by a crank handle P<sup>2</sup> secured to a 75 shaft P<sup>3</sup> journaled in lugs carried at the outer side of the side frames and provided at its rear end with a crank arm P4 having a pin P<sup>5</sup> working in a slot P<sup>6</sup> formed in the arm P' secured to the end of the shaft P'. 80 By turning the handle P2, the belt shipper P may be moved toward and away from the framework D to shift the driving belt, not shown, from the pulley B<sup>2</sup> fast on the shaft B to the pulley  $B^3$  loose on the shaft B.

The operation of the mechanism disclosed is as follows: Assuming the initial position of the apparatus to be that shown in Fig. 3, the plungers H are at the upper limit of their movement and their upper ends are 90 substantially flush with the upper surface of the die block G<sup>1</sup>, which is covered by the material in the shoe K. The rotation of the wheel C<sup>1</sup> in the direction indicated by the arrow causes the cam roll J3 to move into the 95 main circular portion of the cam track C<sup>3</sup>. This draws the plungers H to the lower limit of their movement where the holder H1 engages and is supported by the abutment H5. The plungers H fit snugly in the passages 100 G<sup>3</sup> and as a result, the suction occurring when the plungers H move downward insures that the material to be molded shall follow the plungers into the upper ends of the recesses G<sup>3</sup>. Shortly after the follower 105 H¹ engages the abutment H⁵, the shoe K is given a to and fro movement by the engagement of the cam roll L3 with the hump L5 of the cam L<sup>4</sup>. This assists in completely filling the passages above the plungers H 110 with the material to be compressed, and in compacting the material somewhat.

It will be observed that the front wall of the cavity in the shoe K is inclined at K' where the corresponding portion K\* of the 115 rear wall is vertical. As a result, when the shoe is moved away from the front of the machine, the wall K<sup>\*</sup> tends to wedge down the material in the shoe while on the return movement of the shoe the vertical wall K<sup>5</sup> 120 has no tendency to lift the material in the

shoe.

Shortly after the cam roll L³ leaves the hump L<sup>5</sup> it engages the main circular portion of the cam L4 and the shoe is moved 125 back out of line with the die block G1. During the portion of the operation just described, the wrist pins C<sup>2</sup> are moving through the extreme upper portion of their circle of movement and the plungers F are held well 130

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above the table G. As the rotation of the shaft C continues, the crosshead E descends and the plungers F enter the recesses G³ and complete the formation of the tablets by 5 compressing the material in the recesses G<sup>3</sup>

against the plungers. H.

It will be observed that the mechanism permits of a very powerful and exact pressing action, as the upper portion of the bear-10 ings for the shaft C are formed in the framework D and in the same plane with the plungers, and the holder H is supported on the posts D<sup>2</sup> rigidly connected to the framework D so that there can be practically no 15 injurious lost motion in the apparatus. Moreover the parts are so arranged that the pull on the crosshead E is in practically the same plane as the lines of movement of the compressing plungers, thus avoiding lateral 20 strains.

By adjusting the upper plunger holder in the crosshead E, any desired thickness of tablet may be obtained, and in practice I have found that the machine will make 25 tablets of a thickness varying from that of a sheet of paper up to any thickness within the limits of the machine. The tablets produced with any one adjustment are of uniform quality and are made with very slight

30 waste.

After the tablets are compressed as desired, the parts return to the position shown in Fig. 3. The cams C<sup>3</sup> and L<sup>4</sup> are so shaped and arranged with respect to the cam rolls L<sup>3</sup> 35 and J<sup>3</sup> that the plungers H are moved to the upward limit of their movement just before the shoe K starts to move toward the front of the machine, so that when the shoe K does move to the front of the machine, it 40 moves the tablets off the tops of the plungers H and discharges them into the chute N. The abutment screws H<sup>8</sup> insure that the upper ends of the plungers H shall not project above the surface of the guide block G1, 45 which might otherwise occur from the momentum of the parts and the lost motion incident to use. The arrangement of the abutment screws H<sup>8</sup> so that they project upward from the arms H<sup>9</sup> at the sides of the 50 plunger carrier H insures against any liability of interference in the formation of tablets of the desired thickness by a collection of wastage of material compressed on the upper ends of the abutment screws. By 55 adjusting the block I2 along the arm J, the throw of the lever H<sup>2</sup> and consequent movement of the holder H<sup>1</sup> is adjusted as desired to vary the amount of material drawn into the passages G<sup>8</sup>. After the block I<sup>2</sup> is ad-60 justed, the length of the link formed by the members I and I<sup>1</sup> is adjusted to bring the upper ends of the plungers H flush with the upper surface of the die block G1 with parts in the position shown by the drawings. The 65 abutment screw H5 is then adjusted in the land out of the upper ends of said passages, 130

clamping nut H<sup>7</sup> to the proper height. Similarly the holder F<sup>1</sup> is adjusted in the crosshead E to give the desired thickness of tablet. By adjusting the block K<sup>2</sup> on the rod K1, wear of the parts may be compen- 70 sated for so that the shoe K will always

have the proper movement.

The feed of the material from the hopper M¹ to the shoe K may be varied by changing the height of the hopper M<sup>1</sup> above the table 75 G and also by moving the hopper toward and away from the posts D<sup>2</sup> and thereby changing the angle of the tube M. This adjustment is of importance in obtaining the proper feed by gravity of different materials, 80 some of which feed more freely than others.

Having now described my invention, what I claim as new and desire to secure by Let-

ters Patent is,

1. In a tablet compressing machine, a die 85 block having a passage formed in it, an upper plunger movable into and out of the upper ends of said passages, a coöperating lower plunger entering the lower end of said passage, a plunger carrier to which the lower 90 plunger is rigidly secured, adjustable means for giving reciprocating movements of regulated lengths to said carrier, said carrier having rigid laterally extending arms, vertical abutment engaging screws adjustably car- 95 ried by said arms and extending upwardly therefrom at the sides of said lower plunger, whereby liability of waste material from the compressing operation collecting on the upper ends of said abutment screws is avoided, 100 and abutments carried by the frame work of the machine in position to be engaged by the abutment screws and thereby positively check the upward movement of the lower plunger.

2. In a tablet compressing machine, a die block having passages formed in it, plungers entering the passages of the die block at their lower ends, other plungers movable into and out of the upper ends of said passages, 110 a feed shoe movable across the upper ends of the die block, means for moving said feed shoe into and out of the position in which it extends over said die block, means for moving the lower plungers from the position in 115 which their upper ends are substantially flush with the upper surface of the die block to a lower position while the feed shoe is over the die block to draw material from the feed shoe into said passages, adjustable stops for 120 positively limiting the movement of the said lower plungers in both directions, and means for moving the upper plungers into the passages to compress said material into tablets when the shoe is moved out of said position. 125

3. In a tablet compressing machine, a die block having passages formed in it, plungers entering the lower ends of the passages of the die block, other plungers movable into

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means for feeding material to be compressed into the upper ends of said passages, and means for operating the plungers, the means for operating the lower plungers including a 5 reciprocating plunger holder, an oscillating lever engaging said holder for reciprocating the latter, and means for adjusting the amplitude and limits of the oscillation of said lever, abutment engaging devices for limit-10 ing the movements of the holder in both directions, rigidly secured to said holder, and abutments carried by the framework of the machine in position to be engaged by the abutment engaging portions of the holder, 15 said abutments and abutment engaging portions being relatively adjustable to vary the stroke of the holder.

4. In a tablet compressing machine, a die block having passages formed in it, plungers, 20 means for moving said plungers into and out of the upper ends of said passages, other plungers entering the passages of the die block at their lower ends and means for moving said other plungers in said passages, said 25 means including a lever pivoted to the framework of the machine, means for oscillating said lever, operative connections between said lever and said other plungers including a link and a block movably supported on 30 said lever to which the link is pivotally connected, and means for adjusting said block along said lever to carry the block from the position in which the pivotal connection between it and the link has its axis in line with 35 the pivotal axis of the lever to other posi-

tions in which the axis of the pivotal connection between the link and the block is at one side of the pivotal connection between the lever and the framework.

5. In a tablet compressing machine, a die block having vertical passages formed in it, plungers entering the passages formed in the die block, at their lower ends, other plungers movable into and out of the ends of said pas-45 sages, a feed shoe movable across the upper ends of the die block, means for moving the lower plungers from a position in which

their upper ends are flush with the upper surfaces of the die block to a lower position 50 for moving the upper plungers into and out of the passages and for giving the to and fro movement to the feed shoe, said means including a shaft transverse to, but in the same plane with the line of movement of the 55 plungers, a driving wheel carried by said shaft and extending in a plane including the line of movement of said plungers, said cam

engaging the cams at one side of the wheel 60 for operating the lower plungers, a cam lever engaging the cam at the other side for operating the feed shoe and crank arms carried by said shaft one at each side of the wheel and connections between said crank arms and

65 the upper plungers.

wheel having cams on each side, a cam lever

6. In a tablet forming machine, a reciprocating crosshead, clamping members adjustably secured to said crosshead one above the other, said crosshead and members having two sockets formed in them one socket being 70 formed partly in the crosshead and partly in the lower clamping member and the other socket being formed in alinement with the first socket and partly in the crosshead and partly in the upper clamping member, an 75 abutinent bolt adjustably secured in the last mentioned socket and a plunger shank secured in the lower socket with its upper end engaging said abutment.

7. In a tablet forming machine, a recipro- 80 cating crosshead, clamping members adjustably secured to said crosshead one above the other, said crosshead and members having two sockets formed in them, one socket being formed partly in the crosshead and partly 85 in the lower clamping member and the other socket being threaded and in line with the first socket and formed partly in the crosshead and partly in the upper clamping member, a hollow threaded abutment bolt adjust- 90 ably secured in the threaded socket, a plunger shank secured in the lower socket with its upper end engaging said abutment bolt, said shank being formed with a threaded socket in its upper end, and a bolt passing through 95 said threaded abutment bolt and having its lower end screwed into the socket in said shank.

8. In a tablet forming machine, a die block having recesses in which tablets are com- 100 pressed, a feed shoe having a chamber open to said die block and movable to cover and uncover said recesses, a receptacle for the material to be compressed, a flexible conduit through which the material passes by gravity 105 feed from the receptacle to the shoe, and means for varying the position of the receptacle to change the height of the receptacle above the shoe and the inclination of the conduit.

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9. In a tablet compressing machine, a die block having recesses formed in it, a feed shoe for filling said recesses with material to be compressed, said shoe being reciprocated along said block to cover and uncover 115 said recesses and being formed with a material containing chamber open to the block, a portion of the wall of said chamber immediately adjacent said block being inclined to the vertical so that it wedges the material 120 toward the block and into the recesses as the shoes move to uncover the recesses and the opposite portion of the wall being substantially vertical whereby any wedging action on the material is avoided as the shoe moves 125 to cover the recesses.

EMANUEL L. RICHARDS.

Witnesses:

ARNOLD KATZ, D. Stewart.