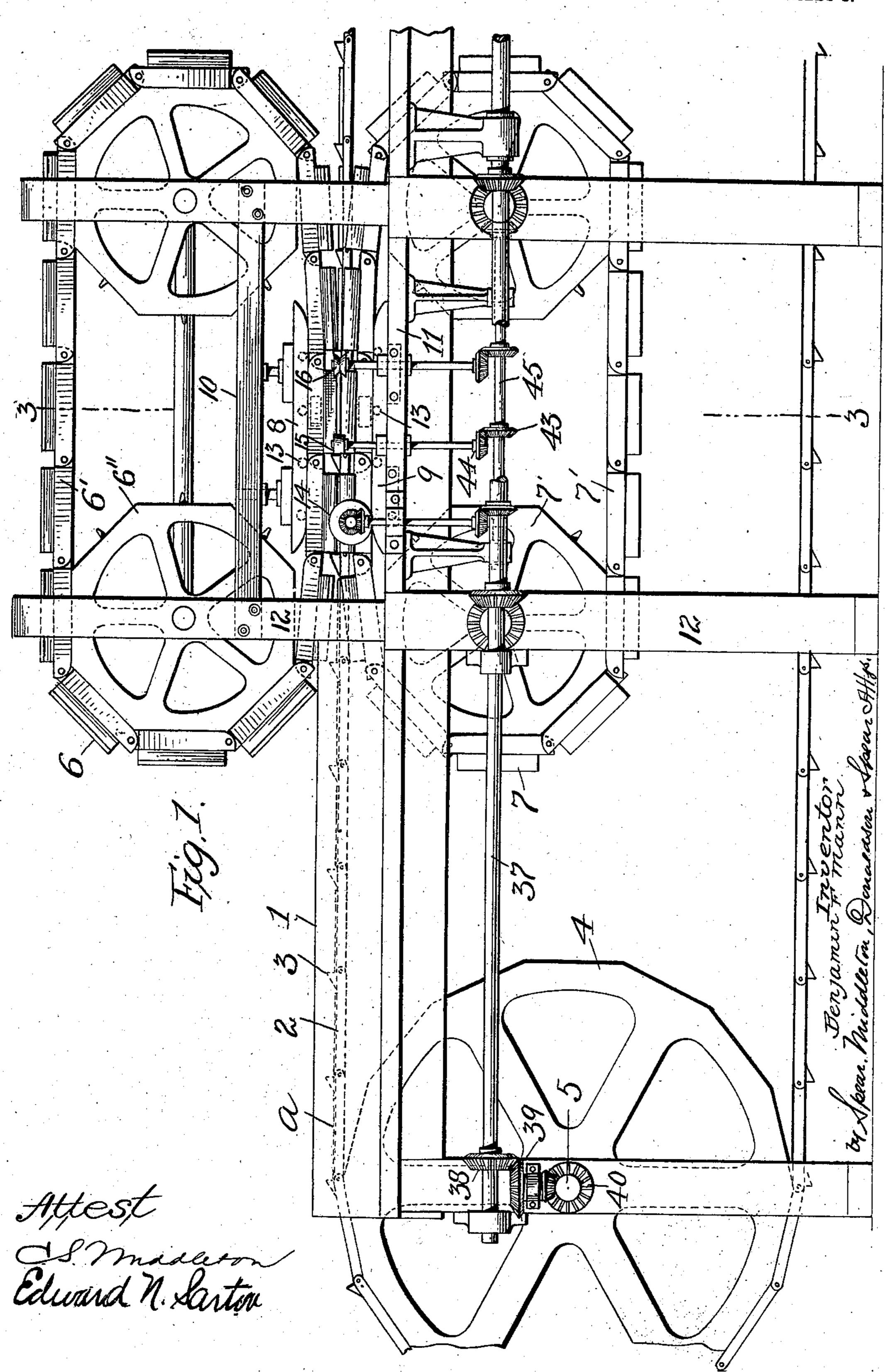
B. F. MANN.

CAN MAKING MACHINE. APPLICATION FILED SEPT. 24, 1906.

5 SHEETS-SHEET 1.

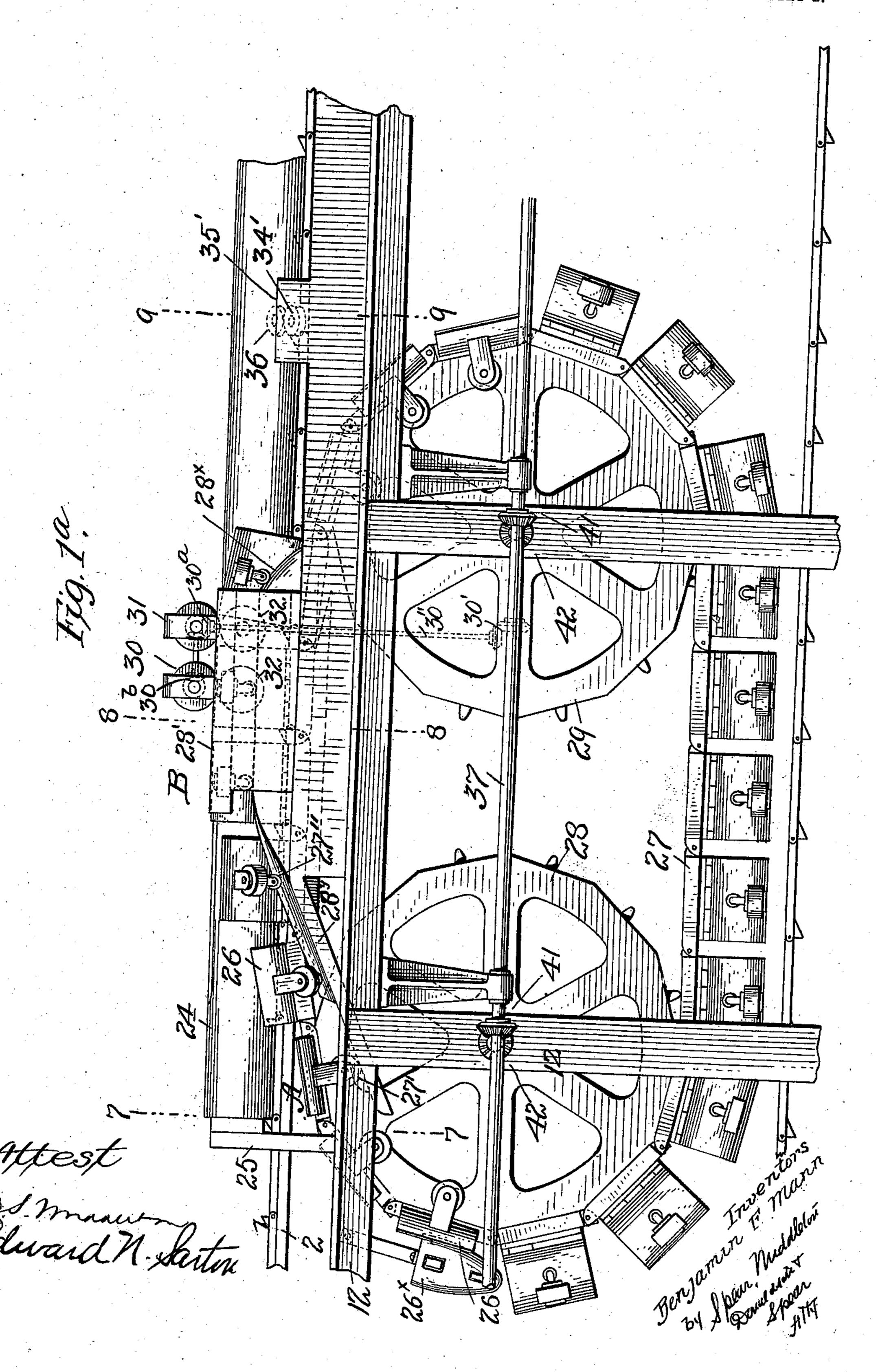


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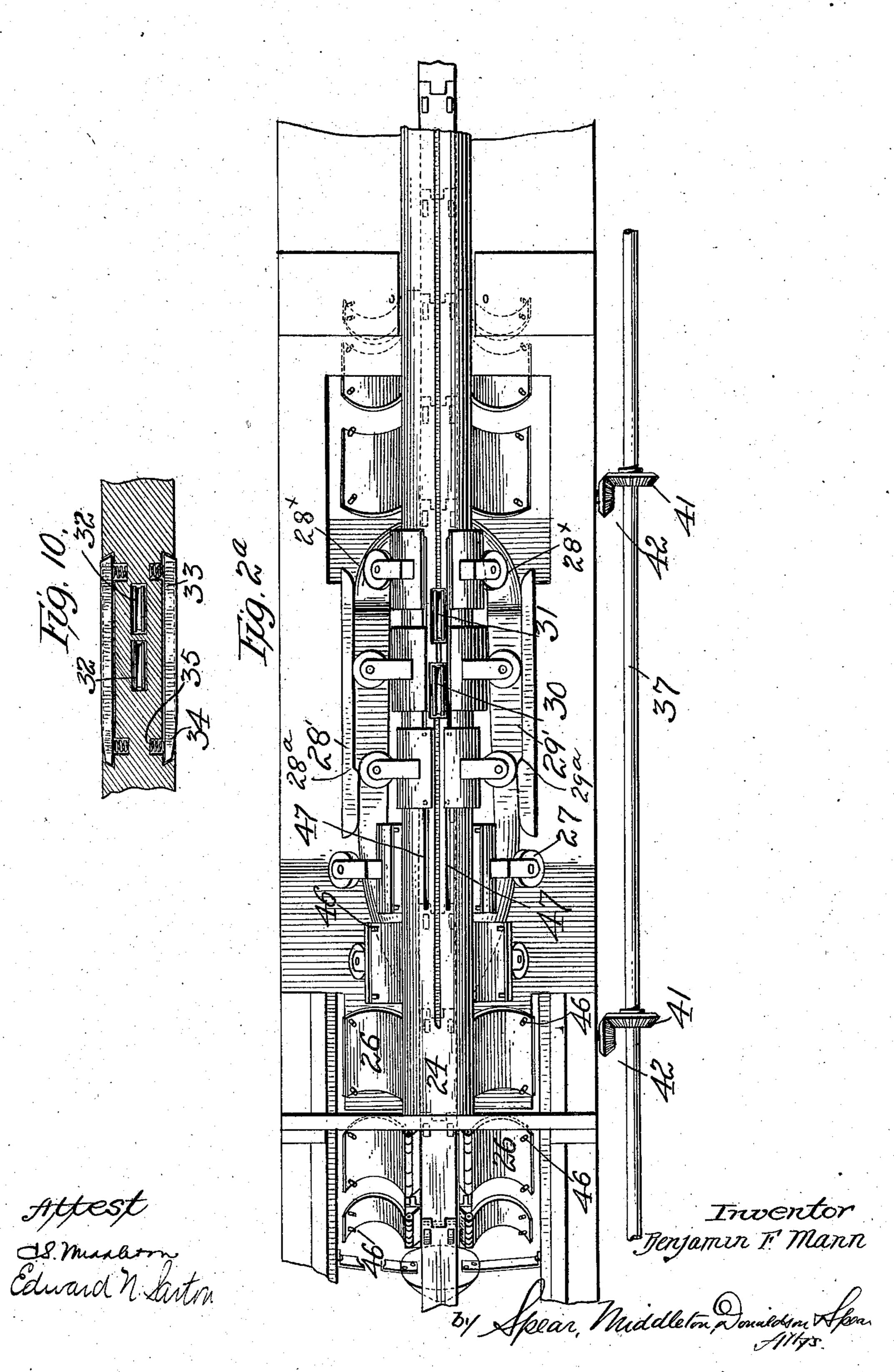


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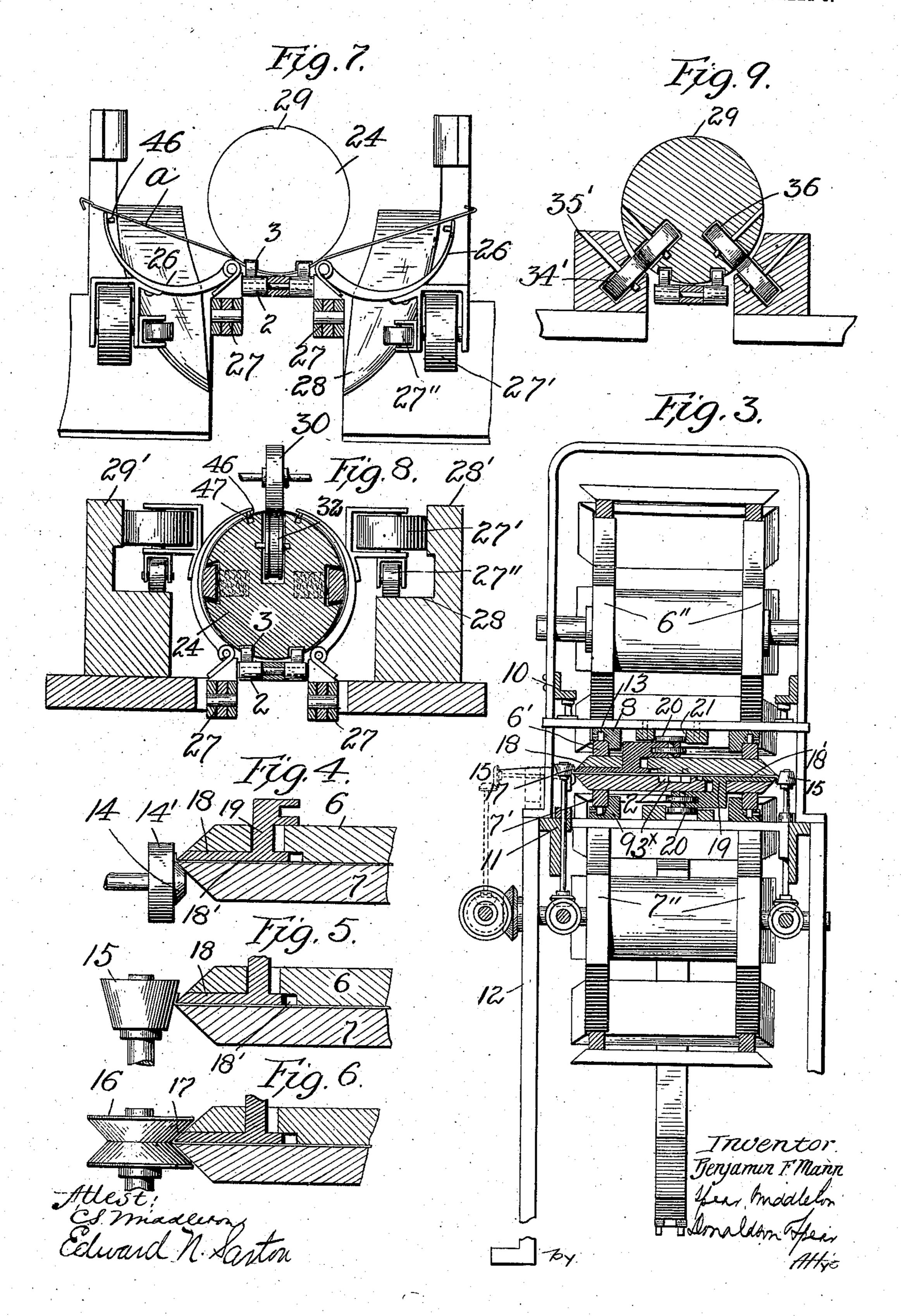


THE NORRIS PETERS CO., WASHINGTON, D.

B. F. MANN.

CAN MAKING MACHINE. APPLICATION FILED SEPT. 24, 1906.

5 SHEETS-SHEET 5.



UNITED STATES PATENT OFFICE.

BENJAMIN F. MANN, OF BALTIMORE, MARYLAND.

CAN-MAKING MACHINE.

No. 881,326.

Specification of Letters Patent.

Patented March 10, 1908.

Application filed September 24, 1906. Serial No. 335,905.

To all whom it may concern:

Be it known that I, Benjamin F. Mann, a citizen of the United States, residing at Baltimore, Maryland, have invented certain 5 new and useful Improvements in Can-Making Machinery, of which the following is a specification.

My invention relates to the manufacture of cans, and particularly to the machinery 10 for making can bodies from previously cut

blanks.

In carrying out my invention I design to provide a machine in which the blanks will be fed in flat form and will pass continuously 15 through the machine and in said continuous passage the edges of the blanks will be bent or flanged so as to interlock, and then the said blanks will be shaped into cylindrical form and the edges interlocked and the seam 20 compressed.

The invention consists in the features and combination and arrangement of parts hereinafter described and particularly pointed out

in the claims.

25 In the accompanying drawings,—Figure 1 is a side elevation of the left hand half of the machine, and Fig. 1^a is a similar view of the right hand half of the machine. Fig. 2 is a plan view, with parts of the frame in section, 30 of the left hand portion of the machine, and Fig. 2^a is a view similar to Fig. 2 of the right hand portion of the machine. Fig. 3 is a vertical sectional view on line 3—3 of Fig. 1. Figs. 4, 5 and 6 are detail views showing the 35 rollers for flanging the edges of the sheet metal blanks. Fig. 7 is a cross sectional view substantially on the line 7—7 of Fig. 1^a. Fig. 8 is a vertical sectional view substantially on the line 8—8 of Fig. 1^a. Fig. 9 is a 40 vertical sectional view on the line 9—9 of Fig. 1^a, and Fig. 10 is a horizontal sectional view through the horn or former at the point where the edges of the blank are interlocked to form the seam.

Referring to Figs. 1 and 1^a, 1 indicates the feeding-in end of the machine at which the sheet metal blanks indicated at a are placed upon a carrying chain 2 having projections 3 to bear against the edges of the blanks, the 50 said chain passing over a drum or wheel 4 journaled in the frame at 5, the said chain passing also over a similar drum or wheel not shown at the other end of the machine to the right of Fig. 1^a. This chain is given a con-55 tinuous movement and the blanks carried thereby pass between clamping blocks ar-

ranged in two series 6 and 7. Each series of clamping blocks or plates is carried by an endless chain 6' and 7' and these endless chains pass around the drums or wheels 6", 60 7". The clamping blocks or plates 6 are arranged to press upon the upper side of the blanks, while the blocks or plates 7 engage the underside thereof, and the movement of the parts is such that an upper and a lower 65 clamping plate will meet each of the blanks as fed inwardly by the chain 2 and each of the blanks will be securely gripped between

these clamping blocks or plates.

As shown in Figs. 3, 4, 5 and 6, the upper 70 and lower clamping blocks or plates have their lateral edges beveled and the sheet metal blanks have their edges projecting slightly beyond these beveled edges of the clamping plates. The movement of the sev- 75 eral endless chains is such that the clamping plates upon engaging and clamping a blank will move forward in the same direction therewith and at the same rate of speed, and in this movement the links of the chain carry- 80 ing the clamping plates or blocks will move between cam tracks or ways 8—9. These cam tracks are supported by the cross beams 10 and 11 of the frame 12 in which the shafts of the drums or wheels 6", 7" are journaled. 85 The cam tracks 8 and 9 are provided with antifriction rollers 13 to bear on the links of the chains for relieving the parts of friction. The cam tracks press the links of the chains 6' towards each other so that the blocks or 90 clamping plates carried thereby may firmly grip the blanks, while the projecting ends or edges of said blanks are subjected to the bending or flanging action now to be referred to. In order to accomplish this bend- 95 ing action of the edges of the blank, which must be interlocked to form the seam, we provide a series of rollers on each side of the machine, as shown at 14, 15 and 16. These rollers are disposed in the order named in the 100 direction of movement of the blanks so that the projecting edge of the blank at each side of the machine will be first subjected to the action of the rollers 14, then to the action of the rollers 15, and then to the action of the 105 rollers 16, so that as shown in Figs. 4, 5 and 6, the edge of the blank will be gradually turned over and inwardly so as to be at an acute angle to the main body of the blank, as shown in Fig. 6 at 17. In order to accom- 110. plish this action the rollers 14, 15 and 16 are of different shapes, those marked 14 and 15

being conical, but at right angles in respect to each other, while the roller 16 has an angularly grooved periphery. The rollers 14 each

have a flange 14'.

As will be seen from Fig. 3, the projecting edge at one end of the blank is turned upwardly, while the projecting edge at the other end of the blank is turned downwardly and the rollers at the opposite ends of the blanks are relatively arranged to accomplish this action. After the edges of the blank have been bent it is carried forward by the continuous movement of the feed chain 2 and at the point at the right of Fig. 1 where 15 the chains 6' and 7' begin to pass around the right hand drums or wheels the clamps leave the bent or flanged blanks so that they may be continuously fed forward to the forming or bending mechanism. As, however, the 20 bent edges of the blank overlie the beveled ends or edges of the clamping plates, special provision must be made in order to free the clumping plates from the blanks, and for this purpose each clamping plate as provided 25 with a movable section or anvil portion 18, these anvil portions being beveled on their lateral edges to correspond with the bevel of the edges of the clamping plates and furnishing the surface upon which the edges of the 30 blanks are bent. These anvil portions or plates are adapted to have sliding movement on their respective clamping plates, for which purpose they are arranged in recesses 18' in the clamping plates and have portions 35 19 extending through the said clamping plates and overlying the same, these overlying portions being provided with rollers 20 adapted to run in cam tracks 21 carried by the frame, as shown in Figs. 2 and 3. These 40 cam tracks are curved at 22 so that the anvil portions of the clamping plates will be retracted so as to release the bent edges of the blanks previous to the arrival of the clamping plates at the point where they begin to 45 separate by reason of passing around the drums 6", 7". These anvil portions are returned to their normal or outer positions before the clamping plates which carry them reëngage with the blanks, and this return of 50 the anvil portions may be effected by spring members or by forming the forward portion

anvils outwardly. Referring now to Fig. 1a which represents a continuation of the machine shown in Fig. 1, it will be seen that the feed chain 2 carries the blank with its edges bent or flanged from the breaking mechanism shown in Fig. 1 to 60 the former and seaming mechanism shown in Fig. 1^a. This former mechanism comprises a horn or former 24, suitably supported at one end by a standard 25 extending up from the frame 12, said former or horn 65 being grooved on its underside as shown in

of the cam plates 21, as shown at 23 in Fig. 3

so as to engage the rollers 20, and move the

Figs. 7, 8 and 9, for the passage of the feed fingers 3. The flanged blank is fed beneath this former by the chain 2 and in its passage along the former it is subjected to the action of a series of folding wings 26 carried by an 70 endless chain 27 passing around the wheels or drums 28—29, which are journaled in the frame of the machine. These wings are normally opened as they approach the left-hand end of the former or horn in Fig. 1a, and as 75 they move towards the right they are gradually closed. The movement of the parts is such that the folding wings are timed to meet and cooperate with the links of the chain 2, each of which carries thereon a blank with 80 its edges flanged. The folding wings begin to close at about the point A Fig. 1a, and this closing action is due to the roller 27' carried by the folding wings engaging a cam track 28^y supported on the fixed frame of 85 the machine. As the folding wings close a second roller 27" comes into play and when the point B is reached by the folding wings the folding action is completed, and the flanged edges of the blank are in interlock- 90 ing position. The final pressure of the folding wings is derived while they are passing between the upright cam surfaces 28'-29', Fig. 8.

The upper part of the former or horn is 95 grooved in substantially the usual manner as at 29, Figs. 7 and 9, and the compression of the interlocked parts or edges to form the seam is effected as shown in Figs. 1a and 8 by rollers 30 journaled in frame parts 31 and 100 rollers 32 journaled in the horn, the rollers 30 and 32 being arranged opposite each other

as shown.

In order that the parts of the seam may be drawn into firm contact, we provide in the 105 horn, plates or bars 33 having their forward edges beveled at 34 so as to lie within the surface of the former or horn, said plates or bars being pressed outwardly by springs 35. These plates, after the folding wings have 110 folded the blanks and the edges are interlocked, press upon the inner walls of the blanks and force the same outwardly and thus draw the interlocked edges into firm contact, and in this position the seam is 115 pressed down by the rollers 30—31—32. The cylindrical body is now forced along the horn by the action of the body feeding chain 2 and the folding wings begin to separate as they pass around the right-hand 120 drum 29. The right-hand end of the horn or former is supported by angularly arranged rolls 34' supported in brackets 35' of the frame, said rolls 34' bearing upon rolls 36 journaled within the horn or former. 125 This manner of supporting the horn does not interfere with the free passage of the cylindrical blank to the discharge end of the machine.

Any suitable driving mechanism may be 130

employed for the various parts and as a | projecting edges, said clamping means movshaft through a chain of beveled gears 38—39 5 and 40 driving the feed chain drum 4, and also through beveled gearing 41 and 42, driving the folding wing carrier drums 28—29. The drums of the chains 6', 7' may be driven through suitable beveled 10 gearing and the flanging rollers.

43—44 from a countershaft 45 which is geared in any suitable manner to the shaft 37. The rollers 14 may be driven as shown

15 in Fig. 2 through gearing 14[×].

From the above it will be seen that the blanks are carried by a continuous motion through the machine and during their continuous passage therethrough they are 20 flanged at their edges and they are then folded and their edges interlocked. These edges are then compressed to form the seam and the cylindrical bodies are then discharged from the machine.

25 In order to properly aline the blanks for the folding and seam forming operation, we provide the folding wings with projecting fingers or pins, 46, to engage the edges of the blanks and hold them in position. When 30 the folding wings are closed these fingers or

pins enter the grooves 47 in the horn.

It will be noticed from Fig. 3 that the upper faces of the lower clamps are provided with grooves 3^{\times} in which the links of the

35 chain 2 lie. By reference to Fig. 1^a it will be seen that I employ an opening cone 26[×] located at the left of the horn and folding mechanism so as to open the folding wings before they arrive 40 at the position where they engage the blanks. This cone is supported from the frame work in any suitable manner.

I do not limit myself to any particular means for interlocking the flanged edges of 45 the blanks, but I prefer, as shown in Fig. 2a to form depressions at the points 28^a and 29^a in the cam surfaces 28'-29' so that after the folding wings have folded the blank the rollers of the folding wings will move backwardly 50 when they reach the shoulders 28a under the action of the spring pressed plates or bars 33 and thus the interlocking effect is insured. I employ inclines 28[×] at the right of the cam surfaces 28' and 29' for the rollers of the 55 folding wings to run upon. Crimping or pressure rollers 30—31 may be operated through any suitable connections, as for instance the gears 30', the shaft 30", the gears $30^{\rm a}$ and $30^{\rm b}$.

What I claim is:—

1. In combination in a can making apparatus, means for feeding the can blanks, means for clamping the said blanks, leaving their edges projecting, carrier means for the 65 clamping means, and means for bending the

representative form I show a shaft 37 ex- | ing to keep pace with the onward movement tending longitudinally of the frame, said of the blank and the bending means performing its function while the blank is in

motion, substantially as described.

2. In combination in can making machinery, a horn, means for feeding the can blanks along said horn comprising the endless chain and means independent of said chain for folding the can blanks and forming 75 15 and 16 are driven through gearing | the seam as the said blanks are moving along the horn, said means comprising the endless chain of folding wings and the relatively fixed cam means for closing said wings, substantially as described.

3. In combination in can making machinery, a series of upper and lower clamps, independent carrier means for each of said series whereby said upper and lower clamps are caused to approach each other to clamp 85 the blanks between them and then move in unison with the blanks so clamped, and means for bending the edges of the blanks projecting from the said clamps, said independent carrier means then causing the sepa-90 rating movement of the clamps to release the bent blanks, substantially as described.

4. In combination, the upper and lower series of clamps, the endless carrier for the upper clamps, the endless carrier for the 95 lower clamps, and means for bending the projecting edges of the blanks as they are carried along past the bending means by the endless carriers, and while gripped by the said clamps, substantially as described.

5. In combination, the upper and lower clamps, endless carriers therefor, an endless carrier for the blanks, said carriers all moving in unison, and means for bending the edges of the blanks as they are carried along 105 by the carriers, substantially as described.

6. In combination, the clamps, the carriers therefor, the cam tracks for pressing the clamps towards each other, and means for bending the projecting edges of the blanks 110 as they are carried past the cam tracks and bending means, substantially as described.

7. In combination, in can making machinery, a pair of clamps to grip the blanks leaving the edges thereof exposed, means 115 for bending the edges of the blanks and a retractable anvil portion carried by each member of the pair of clamps, said clamps having movement towards and from each other, and each anvil portion having move- 120 ment towards and from the other, substantially as described.

8. In combination, in can making machines, traveling clamping means having retractable anvil portions movable in respect 125 to said clamping means to free themselves from the bent edges of the blanks and means for bending the edges of the blanks, substantially as described.

9. In combination, two members for hold- 130

ing the blank between them with the edges thereof projecting, means for bending said edges in opposite directions, said members lying upon opposite faces of the blank and 5 have movement to clamp the blank and having also retracting movement in opposite directions independently of the clamping and unclamping movements, said retracting movement being substantially at right 10 angles to the clamping movement in order to free themselves from the bent edges of the blank, substantially as described.

10. In combination, the traveling clamps having in addition to their traveling move-15 ment a movement toward and from each other, edge bending means for the blanks, means for moving the clamps, said clamps automatically releasing the blanks at their bent edges by withdrawing therefrom trans-

20 versely, substantially as described.

11. In combination, the traveling clamps, means carrying the clamps, bending means for the blanks, retractable anvil portions carried by the clamps, and cam means for 25 operating the anvil portions, as the clamps are carried past the same by the traveling movement of the clamps, substantially as described.

12. In combination, a horn, means for 30 moving the blanks along the horn, and a series of folding wings with traveling carriers therefor to move the same along the horn in unison with the blanks, and means for giving a folding action to the folding wings, during 35 their traveling movement, said moving means and the traveling carrier for the folding wings being independent of each other, substantially as described.

13. In combination with a horn, the folding wings, fixed cam means for giving pressure to 40 the folding wings, means for pressing the blanks outwardly when folded to interlock the seam, said cams having depressions to allow the folding wings to recede, and traveling carrier means for the folding wings, said 45 traveling carrier means carrying the folding wings past the cam means to be acted on

thereby, substantially as described.

14. In combination in can making machinery, means for feeding the blanks 50 through the machine, clamping means for clamping the blank with its edge exposed, traveling carrier means for the clamping means to keep pace with the onward movement of the blanks, bending means past 55 which the blanks are carried while clamped, folding mechanism and seam forming mechanism, the said feeding means carrying the blanks from the bending and clamping means to the said folding and seam forming mech- 60 anism, substantially as described.

15. In combination, means for feeding the blanks, a series of clamps, endless carriers for the said clamps moving in unison with the feeding means, bending means for the 65 edges of the blanks, and folding and seam forming mechanism, the said blank feeding means delivering the blanks with their edges bent from the clamping and bending means to the folding and seam forming mechanism. 70

In testimony whereof, I affix my signature

in presence of two witnesses.

BENJAMIN F. MANN.

Witnesses:

George R. Crew, CHARLES W. MANN.