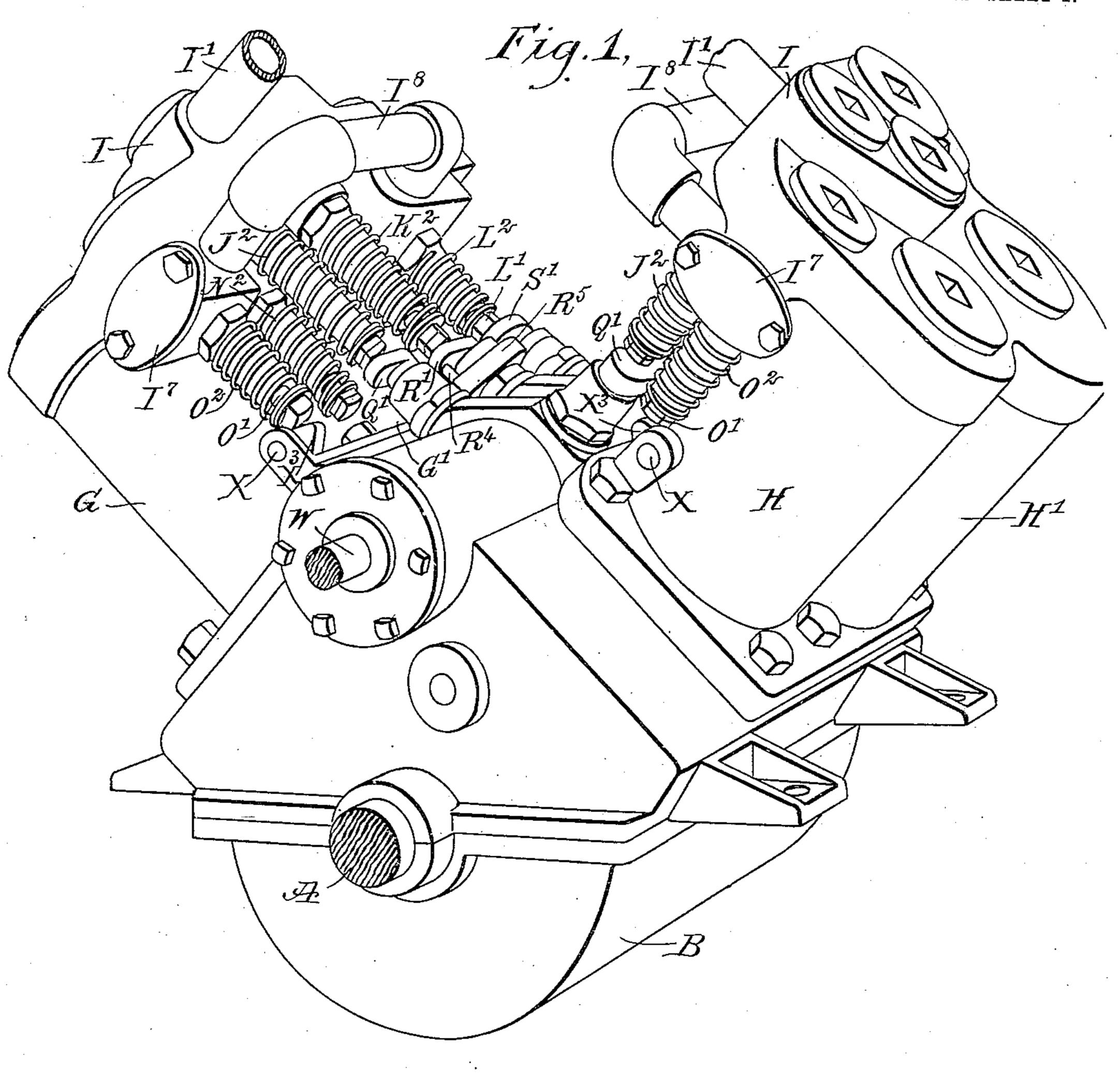
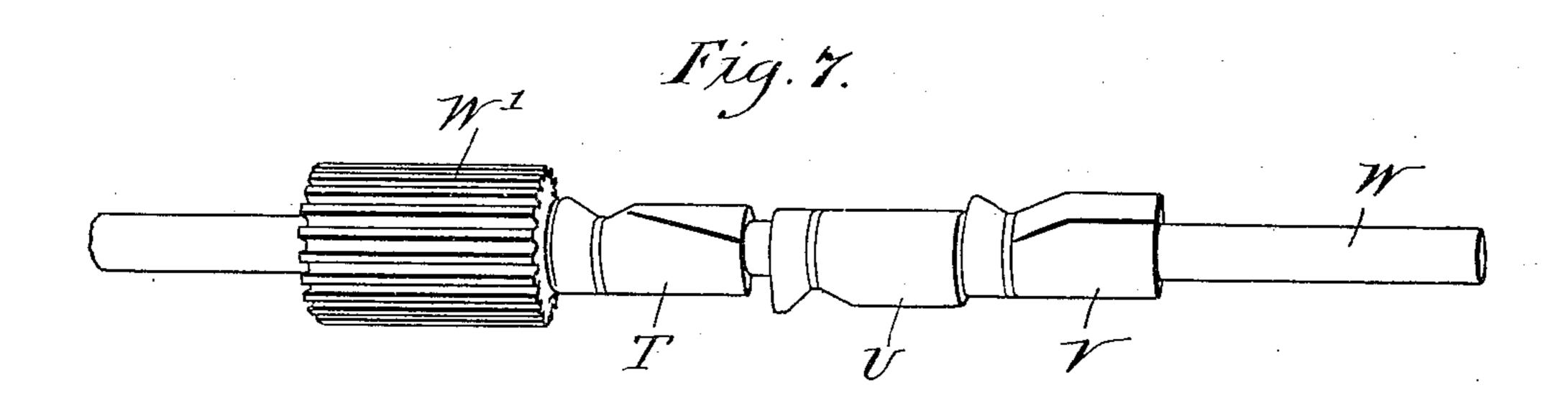
No. 875,453.

PATENTED DEC. 31, 1907.

D. PARRETT. COMPOUND ENGINE. APPLICATION FILED MAR. 7, 1907.

4 SHEETS-SHEET 1.





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Edward Thorpe. Revy. Hoosh 3 Dent Parrett

BY Munn Co

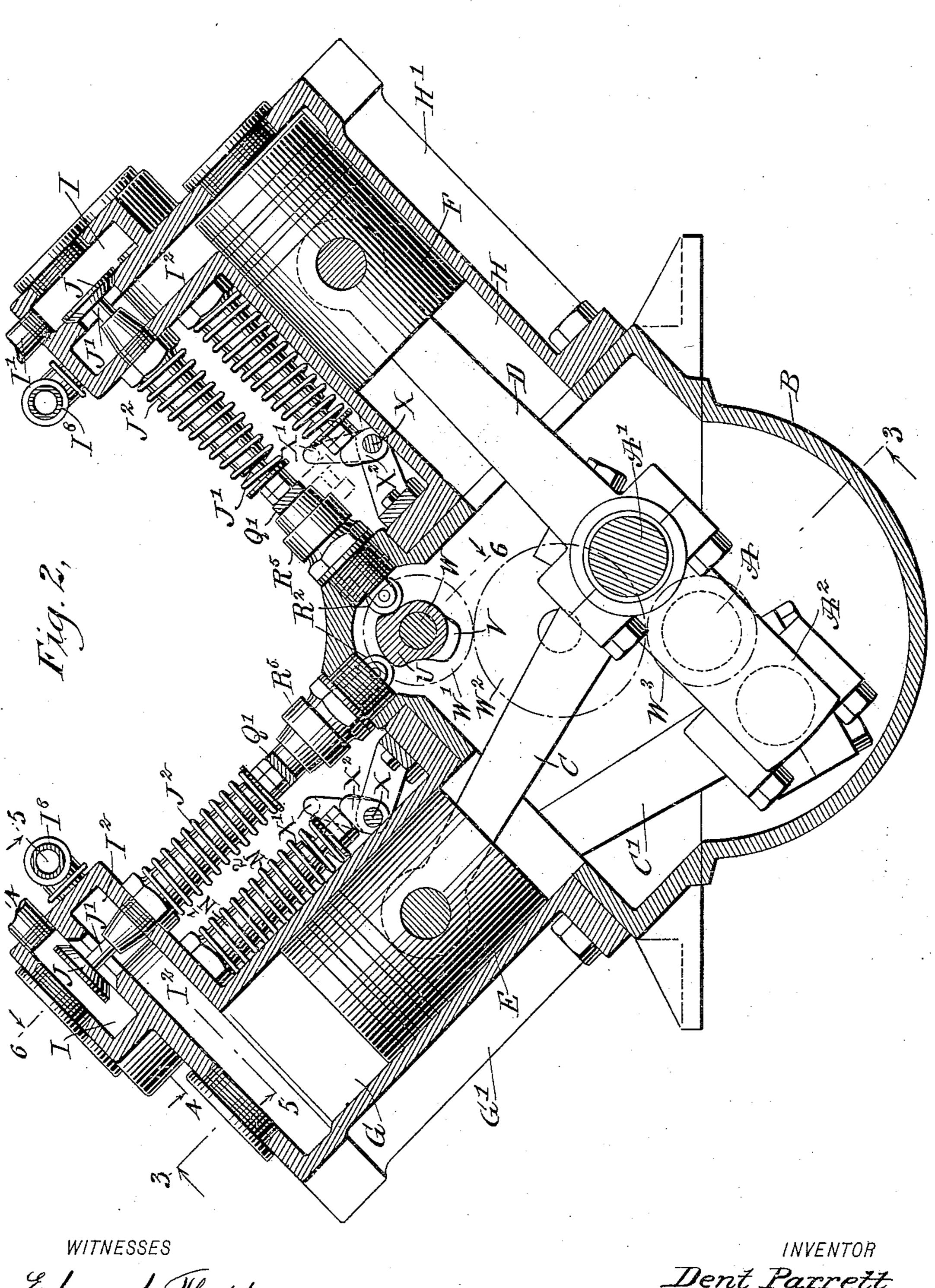
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4 SHEETS-SHEET 2.



Edward Shorpe. Hwy. Hosting. INVENTOR

Dent Parrett

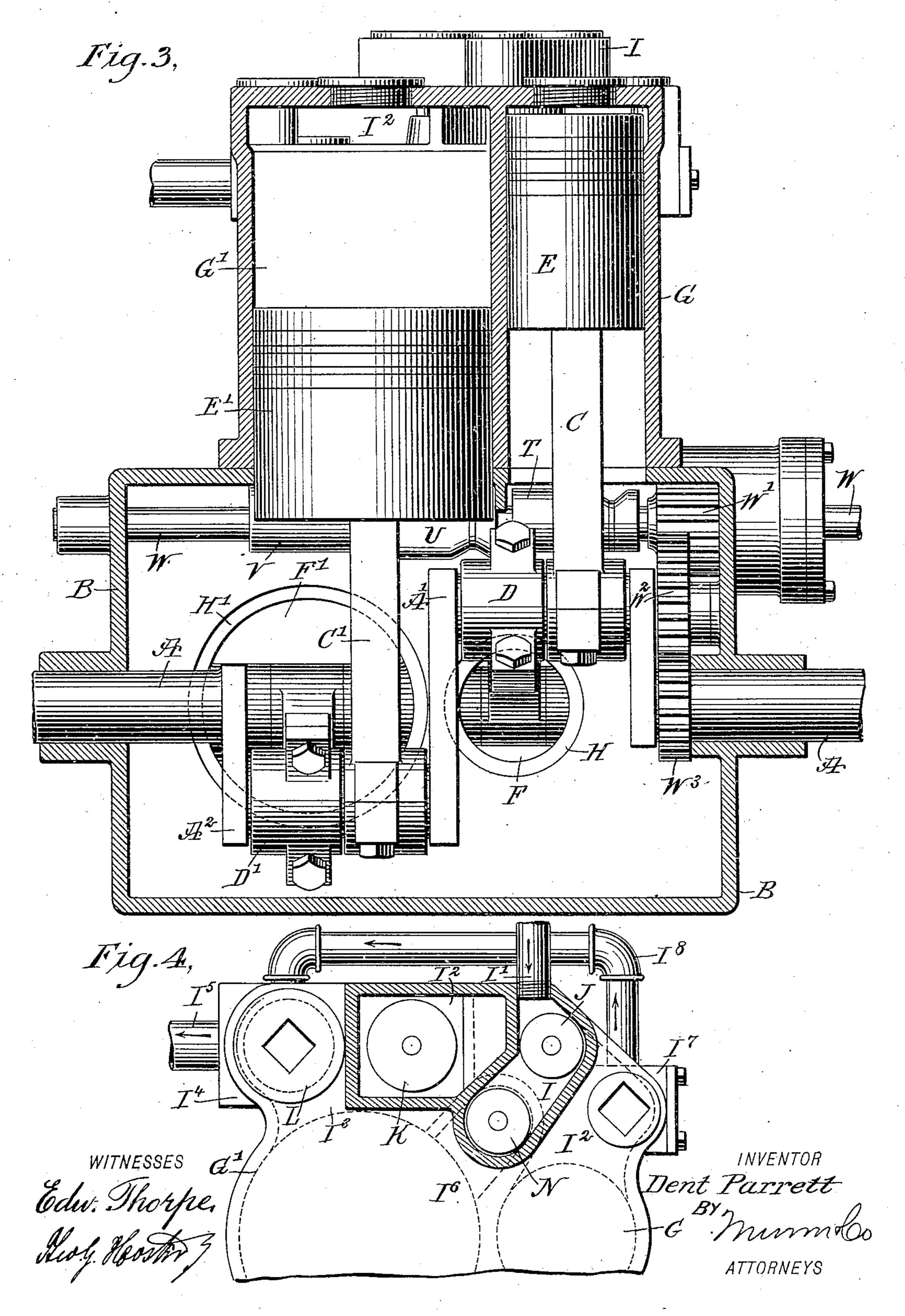
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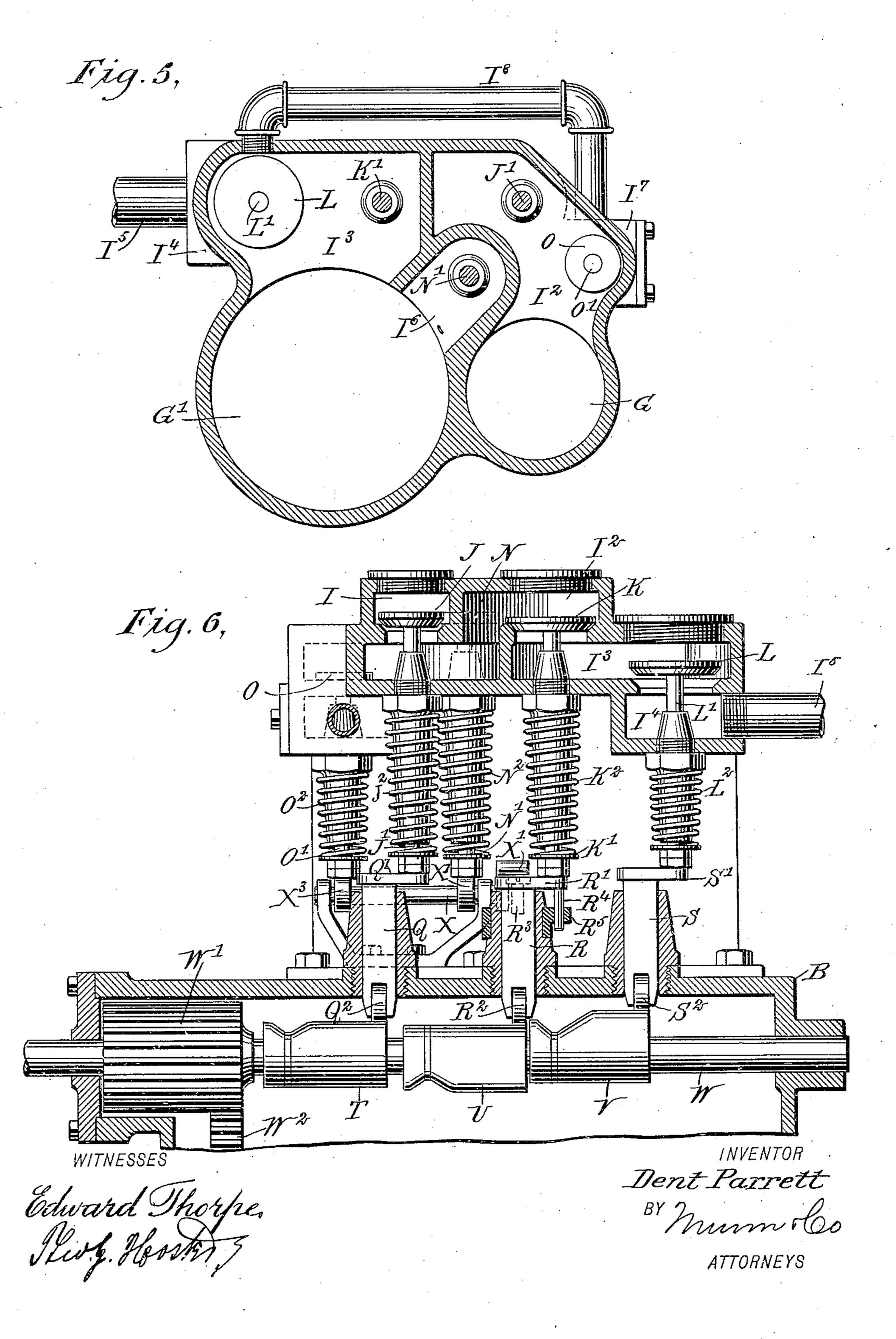
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APPLICATION FILED MAR. 7, 1907.

4 SHEETS-SHEET 4.



UNITED STATES PATENT OFFICE.

DENT PARRETT, OF WENONA, ILLINOIS, ASSIGNOR OF ONE-HALF TO JAMES PARRETT, OF WENONA, ILLINOIS.

COMPOUND ENGINE.

No. 875,453.

Specification of Letters Patent.

Patented Dec. 31, 1907.

Application filed March 7, 1907. Serial No. 361,036.

To all whom it may concern:
Be it known that I, Dent Parrett, a citizen of the United States, and a resident of Wenona, in the county of Marshall and State 5 of Illinois, have invented a new and Improved Compound Engine, of which the following is a full, clear, and exact description.

The object of the invention is to provide a new and improved compound engine, which 10 is simple, durable and compact in construction, completely balanced and hence capable of running at a high speed without danger of undue wear or jar, arranged to start easily by using live motive agent in the low pres-15 sure cylinder, and especially serviceable for use on motor vehicles and the like.

The invention consists of novel features and parts and combinations of the same, which will be more fully described herein-20 after and then pointed out in the claims.

A practical embodiment of the invention is represented in the accompanying drawings forming a part of this specification, in which similar characters of reference indicate cor-

25 responding parts in all the views.

Figure 1 is a perspective view of the improvement; Fig. 2 is a sectional side elevation of the same; Fig. 3 is a transverse section of the same on the line 3—3 of Fig. 2; Fig. 4 is a 30 sectional plan view of the improvement on the line 4—4 of Fig. 2; Fig. 5 is a like view of the same on the line 5—5 of Fig. 2; Fig. 6 is a transverse section of the same on the line 6—6 of Fig. 2, and Fig. 7 is a perspective 35 view of the cams for actuating the admission and exhaust ports.

The crank shaft A of the engine is journaled in the heads of a suitable casing B, and the said crank shaft A is provided within the 40 casing B with the two crank arms A', A2, arranged diametrically opposite each other, and of which the crank arm A' is connected by pitmen C, D with the pistons E, F, reciprocating in the high pressure cylinders G 45 and H, respectively, and the said crank arm A² is connected by pitmen C', D' with the pistons E', F', reciprocating in the low pressure cylinders G', H'.

The high and low pressure cylinders G, G' 50 are arranged one alongside the other and at right angles to the high and low pressure cylinders H, H', which latter are likewise arranged one alongside the other, as plainly indicated in Figs. 1, 2 and 3, the two sets of 55 cylinders G, G' and H, H' being secured to

the casing B and opening with their inner ends into the said casing B. The valve gears for the admission and exhaust of the motive agent are alike for both sets of high and low pressure cylinders G, G' and H, H', so that 60 it suffices to describe but one in detail.

The main steam chest I for each pair of cylinders G, G', H, H', is connected by a pipe I' with a boiler or other suitable source of motive agent supply, and the said main 65 steam chest I is adapted to connect by a valve J with an auxiliary steam chest I2 opening into the outer end of the corresponding high pressure cylinder G or H, so that when the valve J is open the motive 70 agent can pass from the main steam chest I into the auxiliary steam chest I2 and from

the latter into the upper end of the high pressure cylinder G or H, to force the piston E or F in an inward direction, with a view to 75 give an impulse to the main shaft A by the corresponding pitman C or D and the crank arm A'. The auxiliary steam chest I2 is

adapted to be connected by a connecting valve K with a low pressure steam chest I³ 80 opening into the outer end of the low pressure cylinder G' or H', so as to allow the exhaust steam from the high pressure cylinder

G or H to pass into the low pressure cylinder G' or H', to be used expansively therein.

The steam chest I³ is adapted to connect by the exhaust valve L with the exhaust chamber I4, from which leads the exhaust pipe I⁵ to the atmosphere, or to a muffler or other device. The main steam chest I is 90 adapted to connect by a high pressure admission valve N with the high pressure steam chest I6 opening directly into the outer end of the low pressure cylinder G' or H', to allow, whenever desired, live steam to 95 pass from the main steam chest I' into the steam chest I6 and into the low pressure cylinder G' or H', to thus convert the latter into a high pressure cylinder whenever it is desired to start the motor or for other pur- 100 poses. The auxiliary steam chest I2 is adapted to be connected by a valve O with the chamber I⁷ connected by a pipe I⁸ with the exhaust chamber I4, so as to permit a free exhaust of the steam from the corre- 105 sponding high pressure cylinder G or H whenever the low pressure cylinder G' or H' is used as a high pressure cylinder, that is, when the valves N are opened.

The stems J', K', L', N' and O' of the sev- 110

eral valves J, K, L, N, and O are pressed on by springs J2, K2, L2, N2 and O2, to hold the said valves normally to their seats, and the stems J', K' and L' are adapted to be en-5 gaged by the arms Q', R' and S' of slides Q, R and S, mounted to slide in suitable bearings arranged on the casing B, as plainly illustrated in Fig. 6. The slides Q, R and S are provided at their inner ends with friction 10 rollers Q2, R2, S2 in contact with the cams T, U, V secured on a shaft W journaled in suitable bearings arranged on the ends of the casing B; and on the said shaft W is secured a pinion W' in mesh with an intermediate 15 gear wheel W2 meshing with a gear wheel W3 attached to the crank shaft A. Thus when the engine is running and the crank shaft A is rotated then a rotary motion is given to the cam shaft W, which by its cams T, U 20 and V imparts sliding motions to the slides Q, R and S, to lift the valves J, K and L off their seats at the proper time with a view to make the several connections as above described; that is, when the engine is running 25 as a compound or expansion engine, then the valves J are open at the time the pistons E and F of the high pressure cylinders G and H are at the beginning of their down or inward stroke.

The valves K are opened at the time the pistons E and F are at the beginning of their return stroke and the pistons E', F' are at the beginning of their down stroke, so that the exhaust steam in the high pressure cylinders 35 G and H can pass by way of the valves K into the upper ends of the low pressure cylinders G', H', to force the pistons E', F' inward by the expansion of the steam in the said low pressure cylinders G', H'. When the pistons 40 E' and F' reach the end of their inward strokes, the valves L are opened, so as to permit the exhaust steam in the low pressure cylinders G' and H', to pass by way of the open valves L into the chambers I4 and to 45 the exhaust pipes I⁵. The other valves N and O during the time the engine is running as a compound engine are held in a closed position by their springs. When, however, it is desired to use extra power then both sets 50 of cylinders G, G' and H, H' are used as high pressure cylinders, and for this purpose the following arrangement is made: The arms R^{\prime} for lifting the valves K are mounted to turn

running compound the arms R' are in alinement with the stems K' of the valves K, but 60 when the low pressure cylinders G', H' are to be used as high pressure cylinders then the operator turns the sleeves R⁵ so as to move the arms R' from under the stems K' and to move the said arms R' in engagement with 65 crank arms X' held on rock shafts X jour-

on pivots R³ held on their slides R (see Fig.

gaging sleeves R⁵ mounted to turn on the

bearing for the slide R. When the engine is

55 6) and from the arms R' depend pins R4 en-

naled in suitable bearings arranged on the casing B (see Figs. 2 and 6.) On the rock shaft X are secured arms X2, X3 adapted to lift the valve stems N' and O' of the valves N and O, to open the same at the proper 70 time, that is, to allow live motive agent from the steam chests I to pass by way of the valves N into the chambers I⁶ for the steam to pass into the upper ends of the high pressure cylinders G', H' at the time their pistons 75 E', F' are at the beginning of their down strokes. When the pistons E', F' are at their return strokes then the valves L open to exhaust the motive agent from the cylinders G', H' now used as high pressure cylin- 80 ders. The exhaust motive agent from the high pressure cylinders G, H passes by way of the valves O into the chambers I⁷ and by the pipes I⁸ into the exhaust chambers I⁴ and to the exhaust pipes I5 instead of passing by 85 way of the now closed valves K into the low pressure cylinders G', H', as above described, and when the engine is running as a compound engine.

The cam shaft W is adapted to be shifted 90 in an axial direction, and for this purpose the gear wheel W' is of sufficient width to remain in gear with the intermediate gear wheel W² when shifting the shaft W. By reference to Fig. 6, it will be seen that the 95 cams T, U and V are shaped to give more or less rise to the slides Q, R and S, so as to lift the valves J, K and L longer or shorter periods, according to the amount of cut-off desired. By reference to Fig. 6, it will also 100 be noted that the cams are so shaped as to permit of reversing the engine or shifting the cam shaft W to its extreme right hand position. It will also be noticed that the middle cam U does not give as much of a change to 105 the valve K as the cam T gives to the admission valve J, to prevent a too high compression in the high pressure cylinder when running the engine on a short cut-off.

Suitable means may be employed for 110 shifting the sleeve R5 and the cam W, and hence it is not deemed necessary to further illustrate the same.

Having thus described my invention, I claim as new and desire to secure by Letters 115 Patent:

1. An engine having high and low pressure cylinders, means for controlling the admission of steam to the high pressure cylinder, a connecting valve for conducting the 120 exhaust steam from the high pressure cylinder to the low pressure cylinder, a high pressure admission valve for controlling the admission of live steam directly to the low pressure cylinder and means for operating 125 either the said connecting valve or the said high pressure valve.

2. În a compound engine, a high pressure cylinder, a low pressure cylinder, a main steam chest, means for controlling the ad- 130

mission of steam from the main steam chest to the said high pressure cylinder, a connecting valve for conducting the exhaust steam from the high pressure cylinder to the low pressure cylinder, a high pressure admission valve for connecting the said main steam chest with the low pressure cylinder, and means for operating either the said connecting valve or the said high pressure admission valve.

3. An engine comprising a high pressure cylinder, a low pressure cylinder, a main steam chest, an auxiliary steam chest opening into the high pressure cylinder, a valve 15 for controlling the admission of steam from the main steam chest to the auxiliary steam chest, a low pressure steam chest opening into the low pressure cylinder, a connecting valve connecting the auxiliary steam chest 20 with the said low pressure steam chest to allow the exhaust steam from the high pressure cylinder to pass into the low pressure cylinder, an exhaust valve for the low pressure steam chest, a high pressure steam 25 chest opening directly into the low pressure cylinder, a high pressure admission valve connecting the main steam chest with the said high pressure steam chest, and an exhaust valve for the auxiliary steam chest.

4. An engine comprising a high pressure cylinder, a low pressure cylinder, pistons reciprocating in opposite directions in the said cylinders, a crank shaft, pitmen connecting the crank shaft with the said pistons, 35 a main steam chest connected with a steam supply, an admission valve for conducting the live steam from the said main steam chest to the said high pressure cylinder, a connecting valve for conducting the exhaust 40 steam from the said high pressure cylinder to the said low pressure cylinder, an exhaust valve for the said low pressure cylinder, a high pressure admission valve for connecting the said main steam chest with the said low 45 pressure cylinder, and means for operating either the said connecting valve or the said high pressure valve.

5. An engine comprising a high pressure cylinder, a low pressure cylinder, pistons 50 reciprocating in opposite directions in the said cylinders, a crank shaft, pitmen connecting the crank shaft with the said pistons, a main steam chest connected with a steam supply, an admission valve for conducting 55 the live steam from the said main steam chest to the said high pressure cylinder, a connecting valve for conducting the exhaust steam from the said high pressure cylinder to the said low pressure cylinder, an 60 exhaust valve for the low pressure cylinder, a high pressure admission valve for connecting the main steam chest with the said low pressure cylinder, an exhaust valve for the said high pressure cylinder, and means 65 for operating the said high pressure exhaust | valve when operating the said high pressure admission valve for the low pressure cylinder.

6. An engine comprising a high pressure cylinder, a low pressure cylinder, pistons re- 70 ciprocating in opposite directions in the said cylinders, a crank shaft, pitmen connecting the crank shaft with the said pistons, a main steam chest connected with a steam supply, an admission valve for conducting the live 75 steam from the said main steam chest to the said high pressure cylinder, a connecting valve for conducting the exhaust steam from the said high pressure cylinder to the said low pressure cylinder, an exhaust valve for 80 the low pressure cylinder, a high pressure admission valve for connecting the said main steam chest with the said low pressure cylinder, an exhaust valve for the high pressure cylinder, slides having arms for operating the 85 said admission valve for the high pressure cylinder, the said connecting valve and the said exhaust valve for the low pressure cylinder, cams driven from the main shaft for operating the said slides, a rock shaft having 90 arms for operating the said exhaust valve for the high pressure cylinder and the said high pressure admission valve for the low pressure cylinder, and means for operating the said rock shaft from the arm for the slide 95 controlling the said connecting valve.

7. An engine comprising a high pressure cylinder, a low pressure cylinder, pistons reciprocating in opposite directions in the said cylinders, a crank shaft, pitmen connecting 100 the crank shaft with the said pistons, a main steam chest connected with the steam supply, an admission valve for conducting the live steam from the said main steam chest to the said high pressure cylinder, a connecting 105 valve for conducting the exhaust steam from the said high pressure cylinder to the said low pressure cylinder, an exhaust valve for the low pressure cylinder, a high pressure admission valve for connecting the said main 110 steam chest with the said low pressure cylinder, an exhaust valve for the high pressure cylinder, slides having arms for operating the said admission valve for the high pressure cylinder, the said connecting valve and the 115 said exhaust valve for the low pressure cylinder, cams driven from the main shaft for operating the said slides, a rock shaft having arms for operating the said exhaust valve for the high pressure cylinder and the said high 120 pressure admission valve for the low pressure cylinder, means for operating the said rock shaft from the arm for the slide controlling the said connecting valve, and means for shifting the said cams to regulate the cut 125 off and the reversing of the engine.

8. An engine having high and low pressure cylinders, pistons reciprocating in said cylinders, a crank shaft connected with said pistons, a main steam chest, an admission valve 130

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for conducting the live steam from the main steam chest to the high pressure cylinder, an exhaust valve for the high pressure cylinder, a high pressure admission valve for connect-5 ing the main steam chest with the low pressure cylinder an exhaust valve for the low pressure cylinder, a shaft driven from the crank shaft and provided with cams, and mechanism actuated by said cams for simul-10 taneously operating the exhaust valve of the high pressure cylinder and the high pressure admission valve for the low pressure cylinder. 9. A compound engine, comprising two pairs of cylinders, each pair consisting of a 15 high pressure cylinder and a low pressure cylinder, the pairs of cylinders being arranged at right angles to each other, pistons reciprocating in the said cylinders, a crank shaft connected by pitmen with the said 20 pistons, a main steam chest for each pair of cylinders, admission valves for conducting live steam from said steam chests to the high

pressure cylinders, connecting valves for con-

ducting the exhaust steam from the said high

25 pressure cylinders to the said low pressure

cylinders, exhaust valves for the low pressure cylinders, high pressure admission valves for connecting the main steam chests with the low pressure cylinders, exhaust valves for the high pressure cylinders, springs on 30 the stems of said valves to hold the valves normally to their seats, slides provided with arms for operating the admission valves for the high pressure cylinders, the said connecting valves and the exhaust valves for the low 35 pressure cylinders, a shaft driven from the crank shaft and provided with cams for operating said slides, and means for operating the exhaust valves for the high pressure cylinders and the high pressure admission valves 40 for the low pressure cylinders, the said means being actuated from the arms of the slides controlling the said connecting valves.

In testimony whereof I have signed my name to this specification in the presence of 45

two subscribing witnesses.

DENT PARRETT.

Witnesses:
A. L. Webber,
Chas. R. Howe.