No. 845,714.

PATENTED FEB. 26, 1907.

B. F. MAYO.

SOLE PRESSING MACHINE.

APPLICATION FILED MAR. 8, 1906.

5 SHEETS—SHEET 1.

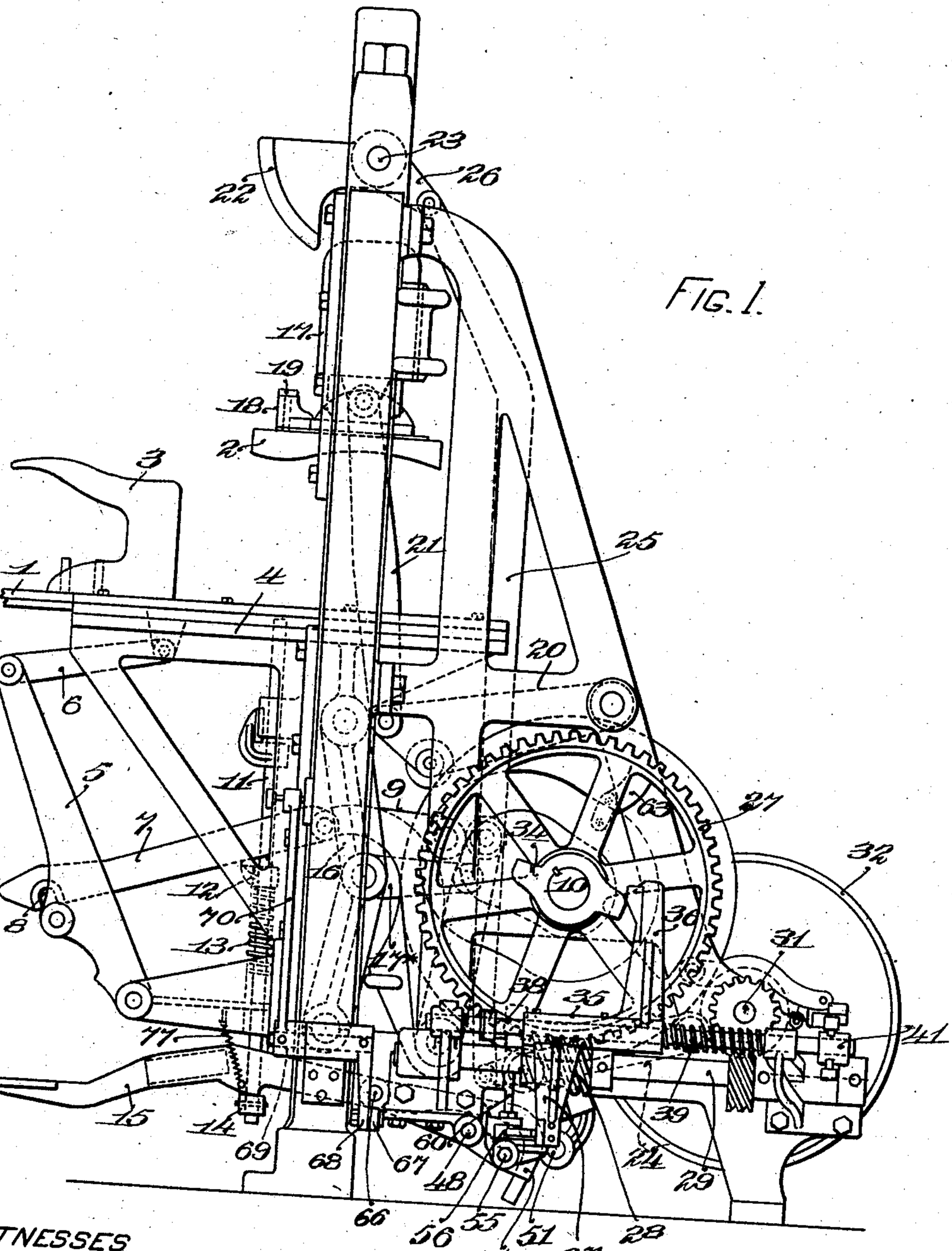

FIG. 1.

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5 SHEETS—SHEET 2.

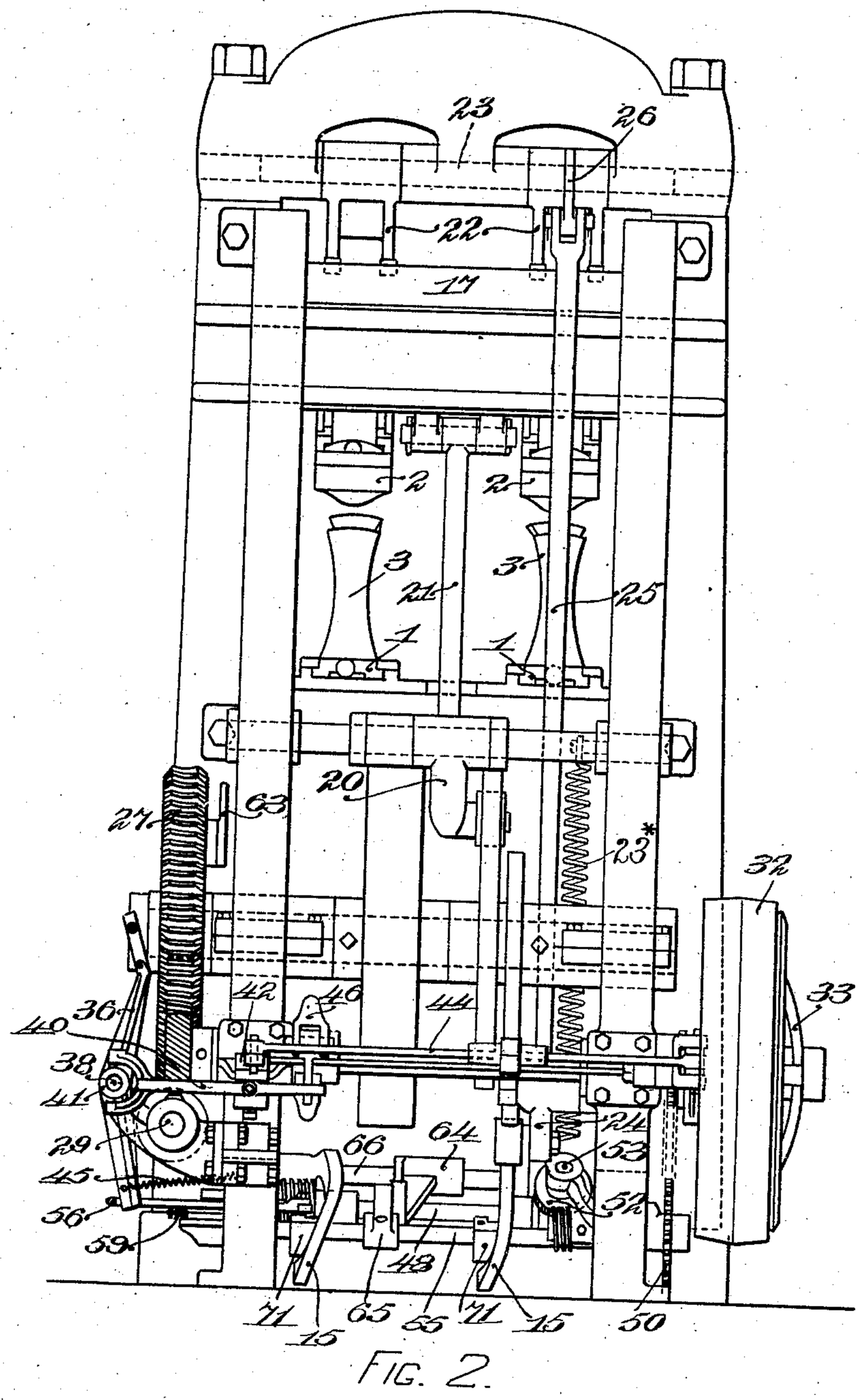

FIG. 2.

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5 SHEETS—SHEET 3.

FIG. 3.

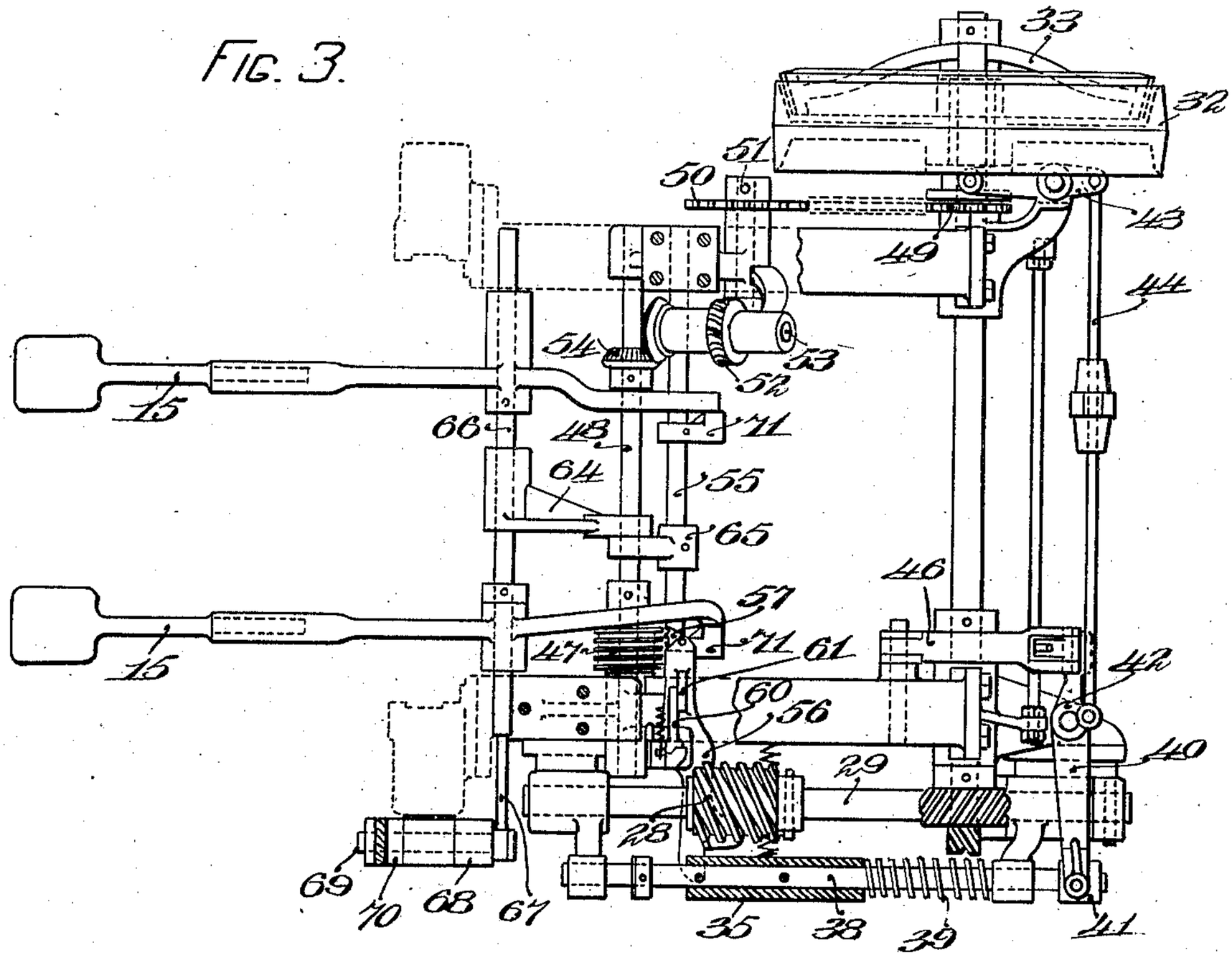


FIG. 4.

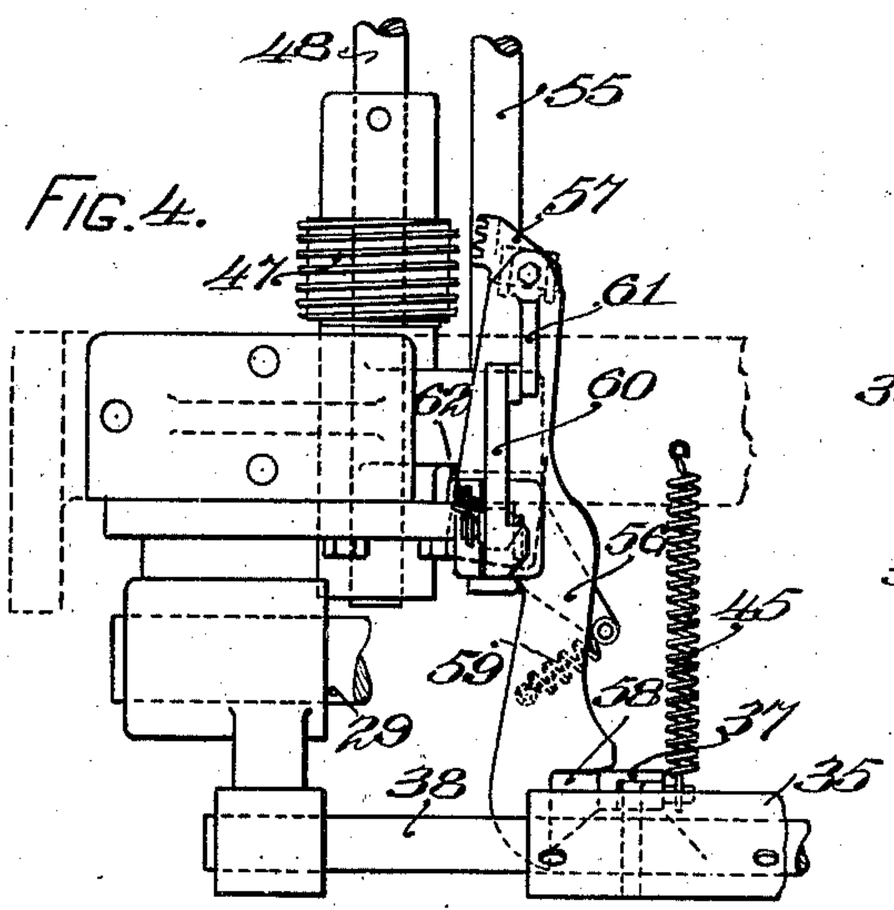
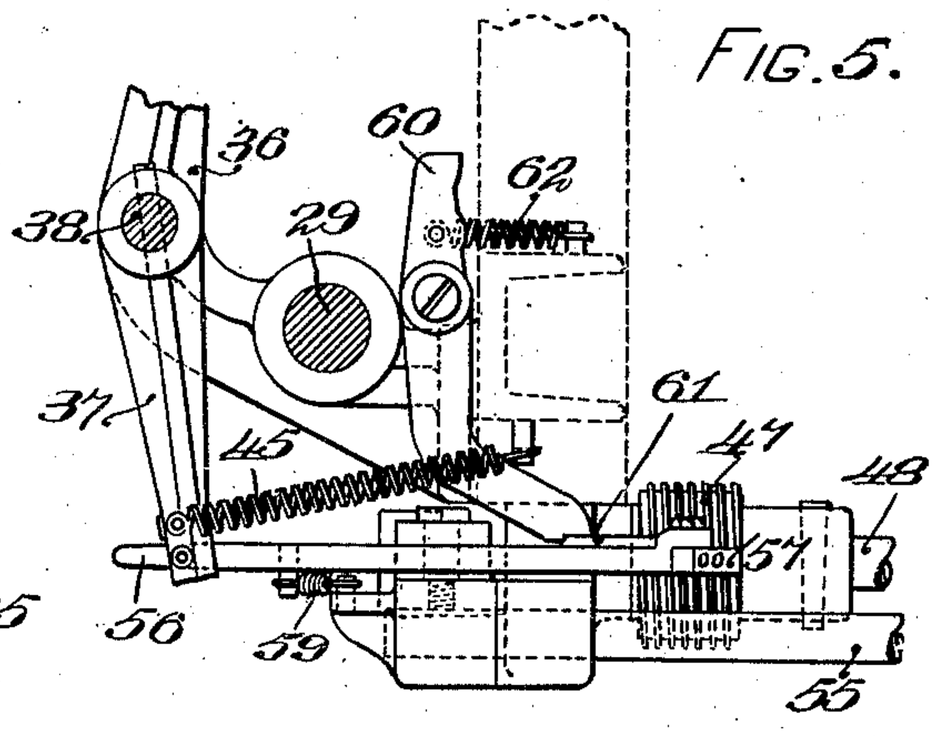


FIG. 5.



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5 SHEETS—SHEET 4.

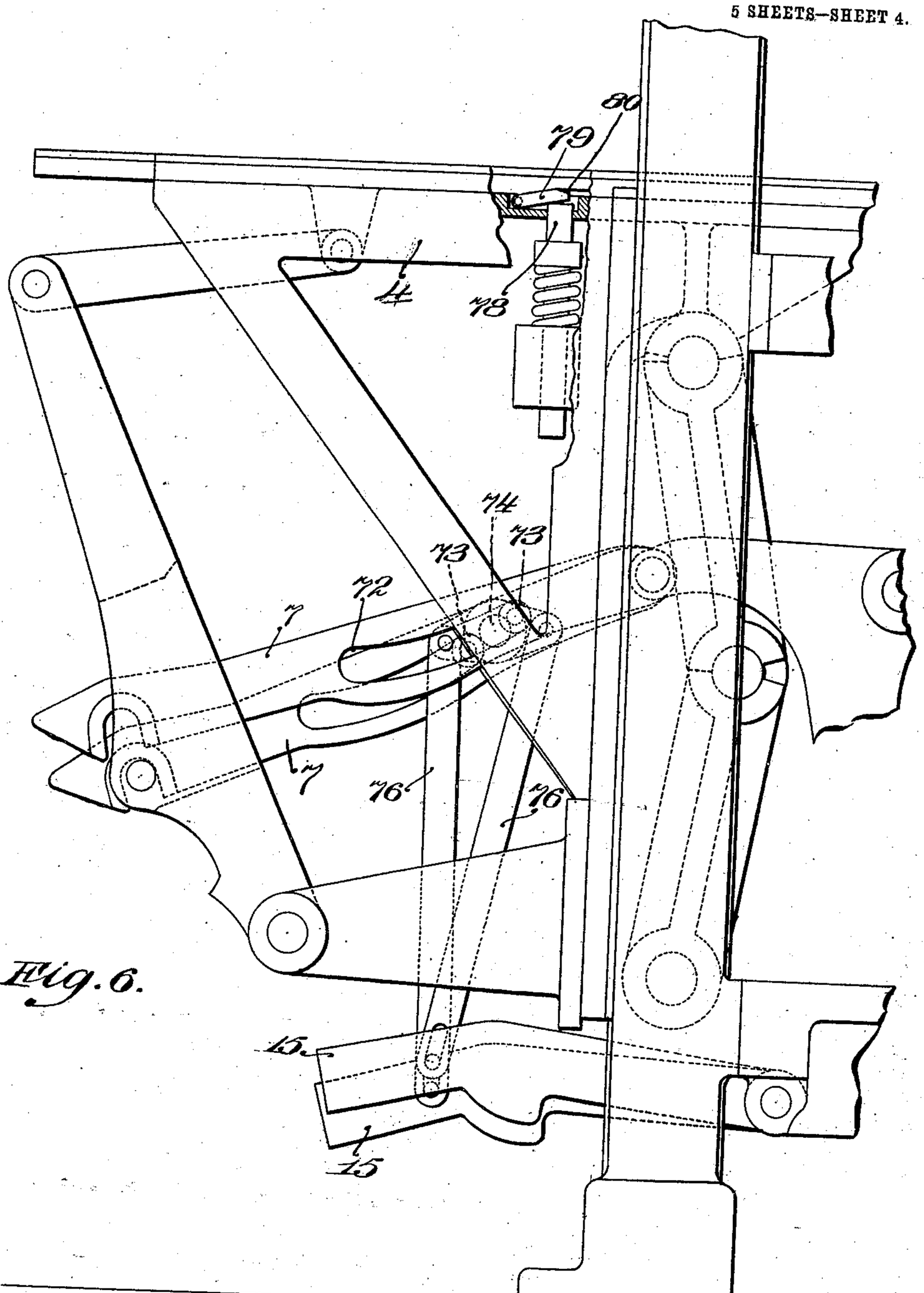

Fig. 6.

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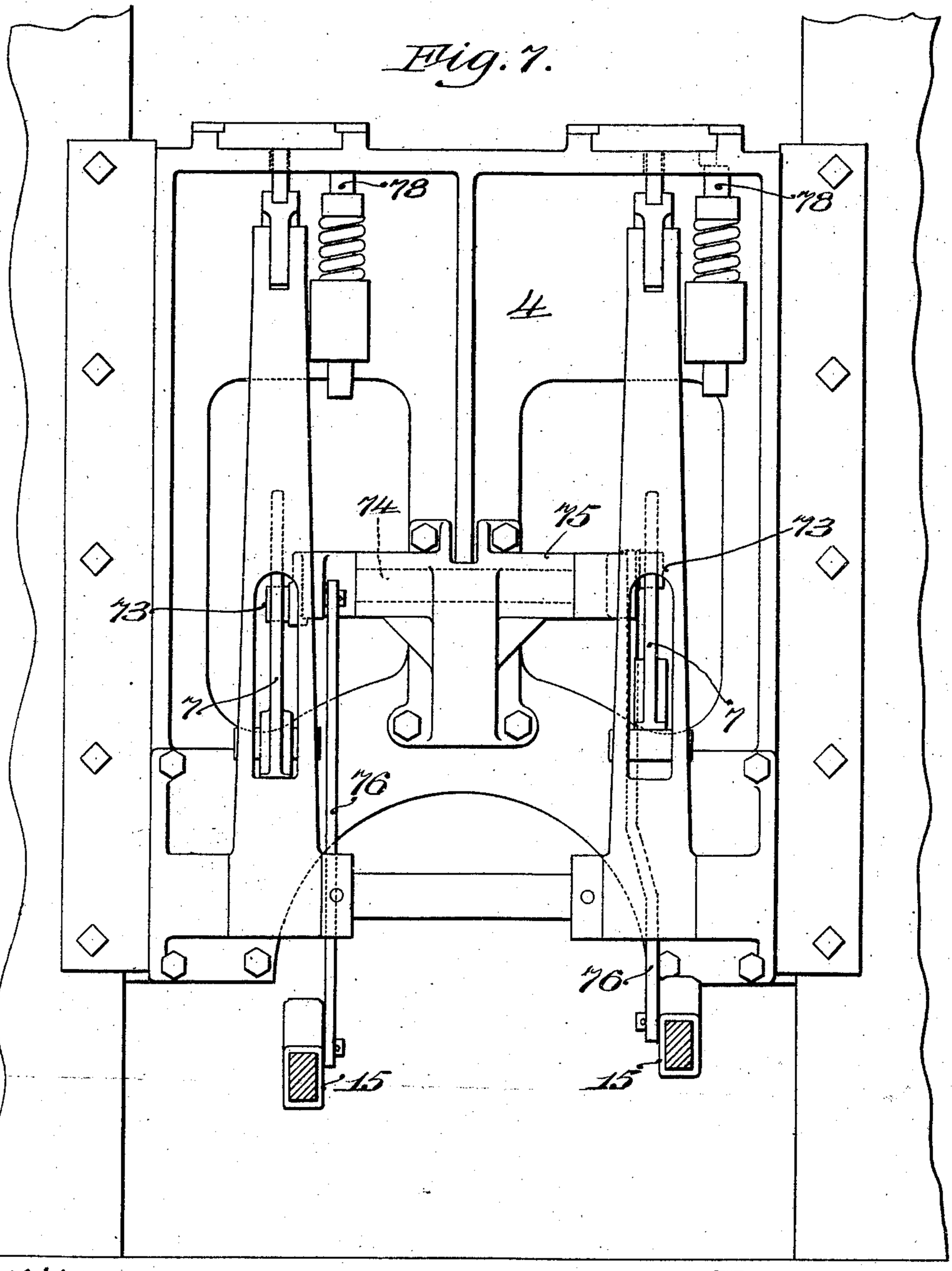
5 SHEETS—SHEET 5.

Fig. 7.



Witnesses
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UNITED STATES PATENT OFFICE.

BENJAMIN F. MAYO, OF SALEM, MASSACHUSETTS, ASSIGNOR TO UNITED SHOE MACHINERY COMPANY, OF PATERSON, NEW JERSEY, A CORPORATION OF NEW JERSEY.

SOLE-PRESSING MACHINE.

No. 845,714. Specification of Letters Patent. Patented Feb. 26, 1907.

Application filed March 8, 1906. Serial No. 304,843.

To all whom it may concern:

Be it known that I, BENJAMIN F. MAYO, a citizen of the United States, residing at Salem, in the county of Essex and State of Massachusetts, have invented certain new and useful Improvements in Sole-Pressing Machines; and I do hereby declare the following to be a full, clear, and exact description of the invention, such as will enable others skilled in the art to which it appertains to make and use the same.

The present invention relates to sole-pressing machines of that type which comprise a plurality of shoe-supporting jacks and coöperating forms and in which means are provided for connecting and disconnecting the actuating mechanism and each jack and its coöperating form in order that a jack may always be in a position of presentation to allow the operator to remove a shoe which has been operated upon therefrom and to place another shoe thereon.

The object of the present invention is to provide means for actuating the devices by which each jack and its coöperating form is operatively connected to the actuating mechanism, whereby the proper actuation of each jack and form is insured and any liability of the jacks and forms being displaced during the operation of the machine, so as to cause inconvenience to the operator or injury to the lasts or the mechanism of the machine, is avoided.

With this object in view the present invention contemplates the provision in a machine of the type referred to of means under the control of the operator for positively actuating the connecting devices to operatively connect each jack and its coöperating form to the actuating mechanism. By providing means for positively actuating the connecting devices a complete movement of the connecting devices into operative position is insured, and any liability of a jack and form being disconnected from the actuating mechanism until the cycle of operations of the machine has been completed is avoided.

The present invention also contemplates the provision in a machine of the type referred to of means whereby when one of the connecting devices is positively actuated to operatively connect a jack and its coöperating form to the actuating mechanism all the other connecting devices are moved to inoperative position, so that by no possibility can more than one jack and its coöperating form be connected to the actuating mechanism at the same time, and any accidental displacement of the jacks and forms which the operator does not intend to throw into operation is prevented.

In the preferred form of the invention the connecting devices are positively actuated through connections to the starting treadle or treadles, by which the actuating mechanism is thrown into operation. In the machine hereinafter described as embodying the various features of the present invention in their preferred form the connecting devices are in the form of latches, and the latches are so connected that when one latch is in operative position the other latches are held in inoperative position. A starting-treadle is provided for each jack and its coöperating form, and each treadle is connected to a latch in such a manner that each latch can be moved into operative position by the treadle connected thereto and simultaneously all the other latches moved into inoperative position.

In addition to the features of the invention above referred to the present invention consists in certain devices, combinations, and arrangements of parts hereinafter described and claimed, the advantages of which will be obvious to those skilled in the art from the following description.

The various features of the present invention will be clearly understood from an inspection of the accompanying drawings, in which—

Figure 1 is a view in side elevation of a well-known form of sole-leveling machine. Fig. 2 is a view in rear elevation of the machine illustrated in Fig. 1 with certain parts omitted to avoid confusion in the drawing. Fig. 3 is a sectional plan view illustrating the mechanism in the lower portion of the machine-frame by which the cam-shaft is thrown into and out of operation. Fig. 4 is a detail plan view, on an enlarged scale, of a portion of the mechanism illustrated in Fig. 3. Fig. 5 is a view in rear elevation of the mechanism illustrated in Fig. 4. Fig. 6 is a view in side elevation of a portion of the machine illustrated in Fig. 1, showing the application

of the present invention to the machine; and Fig. 7 is a view in front elevation of the parts illustrated in Fig. 6.

The machine illustrated in Figs. 1 to 5, to which the preferred embodiment of the present invention has been applied, is a sole-leveling machine of the so-called "direct-pressure" type, in which a relative vertical movement is imparted to the jacks and forms to produce pressure upon a sole and in which horizontal movements are imparted to the jacks to bring them from a position of presentation at the front of the machine to a position in line with their coöperating forms and to return them to their position of presentation. In this machine the connecting devices for connecting each jack and its coöperating form to the actuating mechanism are arranged to connect the jacks to the mechanism by which they are moved horizontally, each jack when the connecting device associated therewith is in operative position being moved horizontally at the proper times during the operation of the machine, while the otner jack remains in its position of presentation. While, however, the invention has been illustrated as embodied in a machine in which the jacks and forms are actuated as above described, it is to be understood that the invention is not limited to use in a machine in which the jacks and forms are so actuated, but may be embodied in other forms of sole-pressing machines, broadly defined in the claims. It is also to be understood that except where such limitations are expressly stated in the claims the present invention is not limited to any specific construction or arrangement of parts.

The machine illustrated in the drawings is provided with two shoe-supporting jacks and with two coöperating sole-pressing forms, the jacks being indicated at 1 and the forms at 2. Each jack is constructed to receive a shoe-supporting last or follower 3 and is mounted to reciprocate in a horizontal guideway formed in the upper portion of a table 4, mounted to reciprocate vertically in guideways on the main frame of the machine. The jacks when connected to their actuating mechanism are reciprocated on the table 4 by means of levers 5, pivoted at their lower ends to brackets projecting from the lower end of the slide which carries the table 4 and connected at their upper ends to the jacks by means of links 6, a lever and link being provided for each jack and the levers being mounted to move independently of each other, so that one jack can remain in a position of presentation while the other jack is being moved into a position to coöperate with a sole-pressing form and returned to its position of presentation. The levers 5 are oscillated by means of links or latches 7, provided with slots 8 in their forward ends, adapted to engage studs on the levers, and pivotally connected at their rear ends to a lever 9, pivoted upon the frame of the machine and provided with a cam-roll engaging a cam on a cam-shaft 10. Both latches 7 are connected to a lever 9, so that they move in unison, and in order to permit either jack to be moved while the other jack remains in its position of presentation vertical rods 11 are mounted to reciprocate in guideways on the slide which carries the table 4 and are provided with sleeves 12, having projections extending beneath the latches 7. A rod 11 is provided for each latch, and the rods are held normally in a position in which the latches 7 are disconnected from the levers 5 by means of springs 13, coiled around the rods and interposed between the lower bearings for the rods and the sleeves 12. At their lower ends the rods are provided with arms 14, projecting beneath treadles 15, pivoted in the lower portion of the machine-frame and acting when depressed to throw the cam-shaft into operation, as will be hereinafter described. A treadle 15 is provided on each side of the machine, and a depression of a treadle acts to depress the rod 11 on the same side of the machine and permit the latch 7 on that side of the machine to engage a lever 5.

The requisite amount of pressure to level the sole of a shoe supported upon a jack is produced by the upward movement of the table 4 when the jack is in position beneath its operating-form. The table 4 is actuated at the proper times during the operation of the machine by mechanism comprising a cam on the cam-shaft 10, a toggle-lever 16, and a link 17*, connecting the toggle-lever and cam.

To automatically regulate the amount of pressure applied to the sole of a shoe, the forms 2 are mounted upon a cross-head 17 and mechanism is provided for permitting the cross-head to descend until a form is seated upon the sole of a shoe for locking the cross-head in position during the upward movement of the table 4 and for raising the cross-head after the sole of the shoe has been leveled. The forms 2 are removably secured to form-carriers 18, being held in position thereon by means of locking-pins 19, mounted in the form-carriers and engaging holes in the straps secured to the forms, and the form-carriers are supported from the cross-head 17 so as to be capable of a slight rocking movement, as is common in this class of machines. The mechanism for permitting the cross-head 17 to descend and seat a form upon the sole of a shoe and for raising the cross-head after the sole of a shoe has been leveled comprises a cam on the cam-shaft 10, a lever 20, pivoted at its rear end to the frame of the machine and provided at its forward end with a roll bearing against the cam, and a link 21, connecting the front end of the lever and the

cross-head. For locking the cross-head 17 in position during the upward movement of the table 4 locking-cams 22 are provided secured to a rock-shaft 23, mounted in the frame of the machine above the cross-head 17, which cams are adapted to be swung inwardly over the cross-head until they contact therewith and to be swung outwardly to allow the cross-head to be raised. The cams 22 are swung inwardly by the force of a spring 23*, (see Fig. 2,) connected to a lever 24 and are swung outwardly by mechanism comprising a cam on the cam-shaft 10, the lever 24, pivotally mounted at its rear end upon the frame of the machine and provided at its forward end with a roll engaging the cam, and a link 25, connecting the forward end of the lever 24 to an arm 26, secured to the rock-shaft 23 and projecting rearwardly therefrom.

The cam-shaft 10 of the machine is driven from the main driving-shaft 31 by connections comprising a worm-wheel 27 on the cam-shaft, a worm 28, meshing therewith, a shaft 29, to which the worm is secured, and spiral gears connecting the shaft 29 and driving-shaft 31. A driving-pulley 32 is mounted to rotate loosely on the driving-shaft 31, and a friction-clutch is provided for connecting the pulley to the shaft when the machine is to be thrown into operation. The friction-clutch consists of two members, one of which is formed on the pulley and the other of which (indicated at 33) is rigidly secured to the driving-shaft, so as to rotate therewith. To connect and disconnect the driving-shaft and pulley, the pulley is moved longitudinally on the shaft. The clutch-actuating mechanism for moving the pulley comprises a cam 34, secured to the cam-shaft 10 outside of the worm-gear 27, a sleeve 35, provided with an upwardly-extending arm 36 and a downwardly-extending arm 37, a rod 38, mounted to rock and move longitudinally in bearings supported from the frame of the machine, a spring 39, surrounding the rod and interposed between the sleeve 35 and one of the bearings for the rod, a lever 40, pivotally mounted in the rear portion of the frame of the machine and connected at one end to the rear end of the rod 38 by means of pins projecting from the forked end of the lever into a grooved block 41, secured to the rod, an arm 42, rigidly connected to the lever 40, a lever 43, connected at one end to the hub of the driving-pulley by means of pins projecting from its forked end into a groove in the hub, and a rod 44, connecting the other end of the lever 43 and the arm 42. When the machine is at rest, with both of the jacks at the front of the machine in their position of presentation, the parts of the clutch-actuating mechanism are in the position illustrated in Figs. 1 and 3, the arm 36 of the sleeve 35 being engaged by one of the projections on the cam 34 and the rod 38 being held in the position to which it has been moved by the cam against the force of the spring 39. To start the machine, the arm 36 is swung inwardly by the operator through means to be hereinafter described and is thereby moved out of engagement with the projection on the cam 34. As soon as the arm 36 moves out of engagement with the projections on the cam 34 the rod 38 is moved to the left by the spring 39 and through the connections above described the driving-pulley 32 is moved into engagement with the clutch member 33. The driving-pulley remains in engagement with the clutch member 33 until the cam-shaft 10 has made one-half of a revolution, at which time the other projection on the cam engages the arm 36, and the rod 38 is returned to the position indicated in Fig. 1, thereby moving the driving-pulley out of engagement with the clutch member 33. The arm 36 after being moved inwardly out of engagement with a projection on the cam 34 is returned into the path of movement of the other projection by means of a spring 45, acting on the lower end of the arm 37, as best shown in Fig. 5.

To stop the rotation of the driving-shaft as soon as the driving-pulley is disconnected therefrom, a friction-brake 46 is provided, which is actuated from the lever 40, as indicated in Figs. 2 and 3.

The projections on the cam 34 of the clutch-actuating mechanism are so located that the driving-pulley is disconnected from the driving-shaft when a jack and form are in a position of pressure and again when the jack has been returned to its position of presentation. As has been stated, the arm 36 is moved inwardly out of engagement with the cam 34 by the operator in order to connect the driving shaft and pulley when both jacks are in a position of presentation. When a jack and form is under pressure, the arm 36 is also moved inwardly to connect the driving shaft and pulley; but this movement of the arm is produced automatically by means of a timing mechanism driven from the driving-pulley. This timing mechanism is best illustrated in Figs. 3, 4, and 5. Referring to these figures, 47 indicates a spiral cam or worm secured to a shaft 48, journaled in the lower portion of the machine-frame and constantly rotated from the pulley 32 by means of a chain passing over sprocket-wheels 49 and 50, secured, respectively, to the hub of the pulley and a short shaft 51, spiral gears 52 connecting the shaft 51 and a shaft 53 and bevel-gears 54 connecting the shaft 53 and the shaft 48. Parallel to the shaft 48 a rod 55 is mounted in the frame of the machine, so as to be capable of moving longitudinally, and upon this rod is pivotally mounted a tripper in the form of a lever 56, the inner end of which is provided with a series of teeth 57, adapted to mesh with the worm 47, and the outer end of which is provided

with a notch 58, through which the lower end of the arm 37 of the clutch-actuating mechanism projects. The tripper 56 is acted upon by a coiled spring 59, which tends to hold the teeth 57 in mesh with the worm 47. Normally the tripper is held in a position in which the teeth 57 are out of engagement with the worm 47 by means of a latch 60, the lower end of which is held in engagement with a projection 61 on the tripper by means of a spring 62, acting on the upper end of the latch. Upon the worm-gear 27 an arm 63 is secured, (see Figs. 1 and 2,) and this projection is so arranged that it comes into engagement with the upper end of the latch 60 and moves the latch to release the tripper 56 just before or at the time that the cam-shaft is stopped with the jack and form in a position of pressure. As soon as the tripper 56 is released it is moved by the spring 59 to bring the teeth 57 into engagement with the worm 47, and while the teeth remain in engagement with the worm the rod 55 is moved longitudinally, and the outer end of the tripper 56, bearing against the arm 37 of the clutch-actuating mechanism, moves this arm outwardly and the arm 36 inwardly until the arm 36 is removed from engagement with the cam 34 and the clutch-actuating mechanism acts to connect the driving shaft and pulley. When the clutch-actuating mechanism thus acts, the rod 38 and the arms 36 and 37 carried thereby move toward the left, as viewed in Fig. 1, and the arm 37 acts on the tripper 56 to swing it about its pivot to a position in which the teeth 57 are out of engagement with the worm 47, in which position it is locked by the latch 60. As soon as the tripper is disconnected from the worm 47 the rod 55 is returned to its initial position by the action of the spring 45.

The initial position of the rod 55 determines the length of time during which the cam-shaft remains at rest with the jack and form under pressure. To enable this time to be varied as desired, an adjustable stop 64 is provided, which is arranged in the path of movement of an arm 65, secured to the rod 55. The stop 64 is mounted upon the shaft 48 and upon a shaft 66, which forms the pivotal support for the starting-treadles 15, and to enable the stop to be conveniently adjusted by the operator it is connected, by means of a rod 67, to the lower end of an arm 68, secured to a short rock-shaft 69, to which rock-shaft an adjusting-lever 70 is secured. The adjusting-lever 70 is located at the front of the machine in convenient position to be grasped by the operator and is provided at its upper end with a locking-pin which coöperates with a series of holes in an indicator-plate to lock the stop 64 in adjusted position.

To enable the cam-shaft 10 to be thrown into operation by the operator after the driving-pulley has been disconnected from the driving-shaft with both jacks in a position of presentation, the rear ends of the treadles 15 are shaped as illustrated in the drawings to coöperate with blocks 71, secured to the rod 55, the shape and arrangement of the rear ends of the treadles and the blocks being such that a depression of either treadle moves the rod 55 longitudinally a sufficient distance to throw the clutch-actuating mechanism into operation.

In the machine illustrated in Figs. 1 to 5 and above described it will be noted that when a treadle 15 is depressed to throw the cam-shaft into operation the corresponding latch 7 is moved into engagement with a lever 5 solely by the force of gravity, the movement of the latch into operative position being permitted by the downward movement of the rod 11 associated therewith, the sleeve 12 of which normally holds the latch 7 raised. In the use of the machine in actual practice trouble has been occasioned by frequent failure of the latch 7 to move a sufficient distance to firmly engage a lever 5, the latch resting insecurely on the cross-pin on the lever and being disengaged therefrom before the jack has completed its inward movement. As a result the jack is not brought into alinement with its coöperating form, and during the upward movement of the table 4 an excessive pressure is brought upon the jack and form which is often sufficient to break the shoe-supporting follower or last and injure other portions of the machine. Also in the operation of the machine both latches 7 are not always raised completely out of engagement with the levers 5 when the jacks are in their position of presentation, and as a consequence both jacks are occasionally connected to the actuating mechanism when the machine is thrown into operation, or the jack which should remain in a position of presentation is displaced, so that the manipulation of the shoes by the operator is interfered with and the follower or last on the jack is injured during the upward movement of the jack-supporting table.

The present invention is designed, primarily, to overcome these defects in the construction and operation of the machine, the preferred embodiment of the invention being illustrated in Figs. 6 and 7. As illustrated in these figures, each latch 7 is provided with a curved slot 72, which slots are engaged by pins or rolls 73 upon opposite ends of a rock-shaft 74, journaled in a bracket 75, secured to the reciprocating slide, which carries the table 4. The pins or rolls 73 are arranged upon opposite sides of the axis of the rock-shaft, so that a movement of the rock-shaft moves the latches positively in opposite directions. Near its ends the rock-shaft is provided with oppositely-extending arms, which are connected, by means of links 76, to

the treadles 15, the arrangement being such that when one treadle is depressed the shaft 74 is rocked in one direction and when the other treadle is depressed the shaft is rocked in the opposite direction. A depression of either treadle therefore acts to positively move the corresponding latch 7 into operative position and the other latch into inoperative position. Before the cam-shaft can be stopped either with a jack and form in a position of pressure or with both jacks in a position of presentation the treadle 15, which has been depressed, must be raised, and this is accomplished by means of springs 77, connected to the treadle, as indicated in Fig. 1. To allow the treadle which has been depressed to rise without actuating the latches, the links 76 are connected to the treadles by pin-and-slot connections, as illustrated in Fig. 6, which connections afford a sufficient amount of lost motion between the links and the treadles to allow the depressed treadle to rise and also the actuation of the latches by the depression of one of the treadles without producing any movement of the other treadle.

In the machine illustrated in Figs. 1 to 5 the rods 11, in addition to forming a connection between the treadles and the latches, are also utilized to lock the jacks in their position of presentation, the upper ends of the rods entering recesses in the jacks when the jacks reach the limit of their outward movement and being held depressed by contact with the lower surface of the jacks during the inward and outward movement of the jacks. The means for locking the jacks in their position of presentation is thus connected to the means for actuating the latches, and the operations of the locking means and the latches are dependent upon each other. In Figs. 6 and 7 an improved means for locking the jacks in their position of presentation is illustrated, which operates independently of the latches, and this locking means constitutes a feature of the present invention. As illustrated, the locking means associated with each jack consists of a spring-pressed plunger 78, mounted in the slide carrying the jack-supporting table 4. At its upper end the plunger 78 bears against a latch 79, pivotally mounted in a recess in the table 4 beneath the jack. At its inner end the jack is cut away on its under surface, as indicated at 80 in Fig. 6, to form a shouldered recess into which the latch 79 is pressed by the plunger when the jack reaches its position of presentation. The engagement of the latch with the shouldered recess is sufficient to hold the jack in its position of presentation while the operator is performing any operations upon the shoe which may be necessary, but permits the jack to be moved inwardly when connected to its actuating mechanism, the latch being depressed by the lower surface of the jack during its inward movement and being held depressed until the jack is again returned to its position of presentation. The interposition of the pivoted latch 79 between the plunger 78 and the lower surface of the jack avoids undue friction between the plunger and the jack or excessive wear and at the same time provides a simple and efficient means for locking the jack in a position of presentation.

The nature and scope of the present invention having been indicated and a mechanism embodying the various features of the present invention in their preferred form having been described, what is claimed is—

1. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, connecting devices for operatively connecting the jacks and forms to said mechanism, and means controlled by the operator for positively actuating said devices to operatively connect each jack and its cooperating form to said mechanism.

2. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, a connecting device for operatively connecting each jack and its coöperating form to said mechanism, and means controlled by the operator for positively moving any one of said devices into operative position and the other devices into inoperative position.

3. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, connecting devices for operatively connecting the jacks and forms to said mechanism, treadles and suitable connections for throwing said mechanism into operation, and means actuated by the treadles for positively actuating said devices to operatively connect each jack and its coöperating form to said mechanism.

4. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, a connecting device for operatively connecting each jack and its coöperating form to said mechanism, treadles and suitable connections for throwing said mechanism into operation, and means actuated by the treadles for positively moving any one of said devices into operative position and the other devices into inoperative position.

5. A sole-pressing machine, having, in combination, a plurality of shoe-supporting

jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, a latch for operatively connecting each jack and form to said mechanism, treadles and suitable connections for throwing said mechanism into operation and connections between each treadle and a latch for positively moving the latch into operative position.

6. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, a latch for operatively connecting each jack and form to said mechanism, and connections between the latches acting when one latch is in operative position to hold the other latches out of operative position.

7. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and its coöperating form to press the sole of a shoe, a latch for operatively connecting each jack and form to said mechanism, treadles and suitable connections for throwing said mechanism into operation, and connections between the treadles and latches for positively moving any latch into operative position and the other latches into inoperative position.

8. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and form to press the sole of a shoe and to return the jack to its position of presentation, a latch for operatively connecting each jack to said mechanism, and means under the control of the operator for positively moving each latch into operative position.

9. A sole-pressing machine, having, in combination, a plurality of shoe-supporting jacks, a plurality of coöperating sole-pressing forms, mechanism for relatively actuating each jack and form to press the sole of a shoe and to return the jack to its position of presentation, a latch for operatively connecting each jack to said mechanism, treadles and suitable connections for throwing said mechanism into operation and connections between the treadles and latches for positively moving any latch into operative position and the other latches into inoperative position.

10. A sole-pressing machine, having, in combination, a shoe-supporting jack, a coöperating sole-pressing form, mechanism for relatively actuating the jack and form to press the sole of a shoe and to return the jack to its position of presentation, a pivoted latch arranged to engage a shouldered recess on the jack and lock the jack in its position of presentation and a spring-pressed plunger supporting the latch.

11. A sole-pressing machine, having, in combination, a shoe-supporting jack, a coöperating sole-pressing form, mechanism for relatively actuating the jack and form to press the sole of a shoe and to return the jack to its position of presentation, and latches operating independently of each other to connect the jack to said mechanism and to lock the jack in its position of presentation.

In testimony whereof I affix my signature in presence of two witnesses.

BENJAMIN F. MAYO.

Witnesses:

FRED. O. FISH,

FARNUM F. DORSEY.