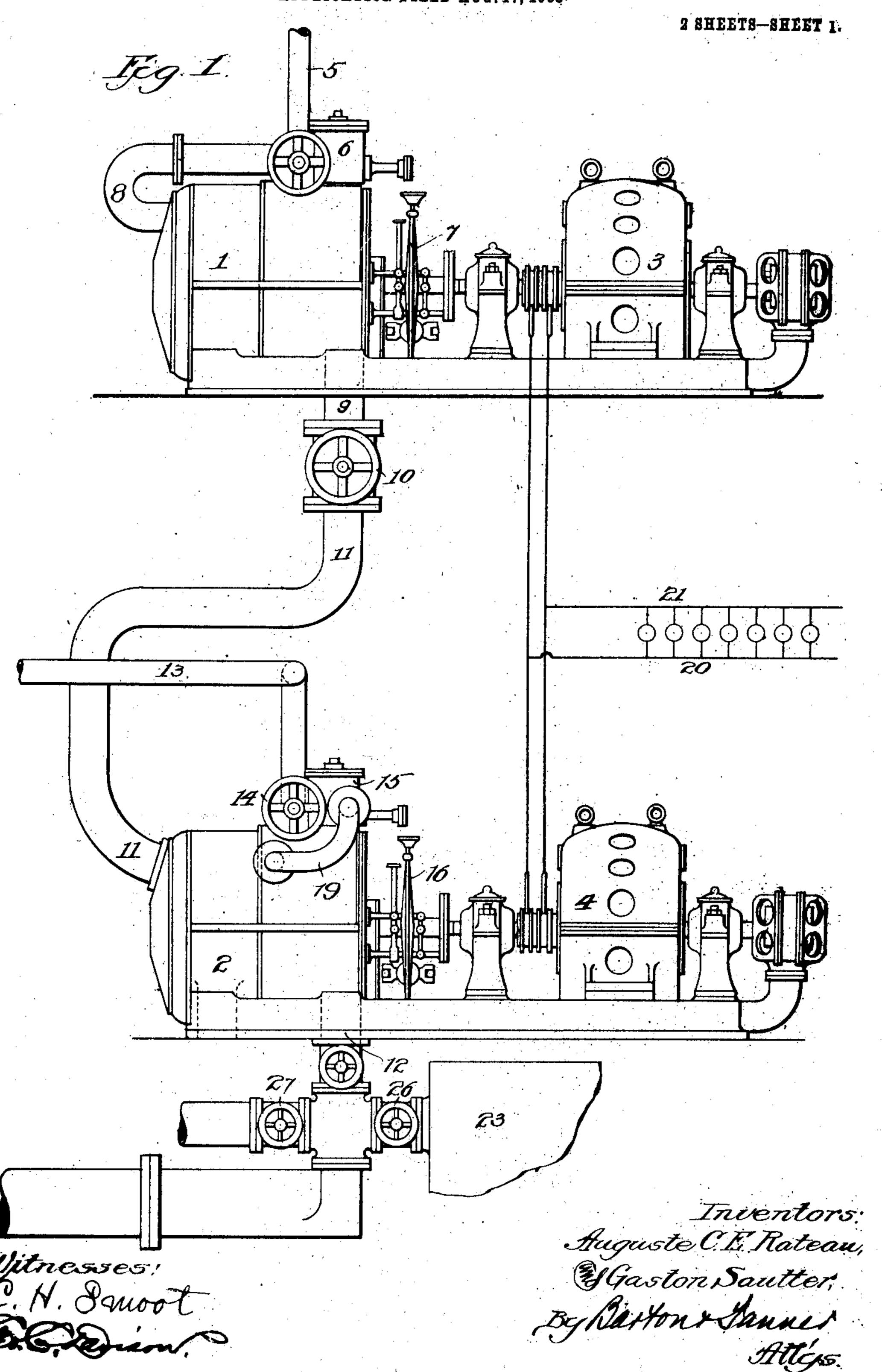
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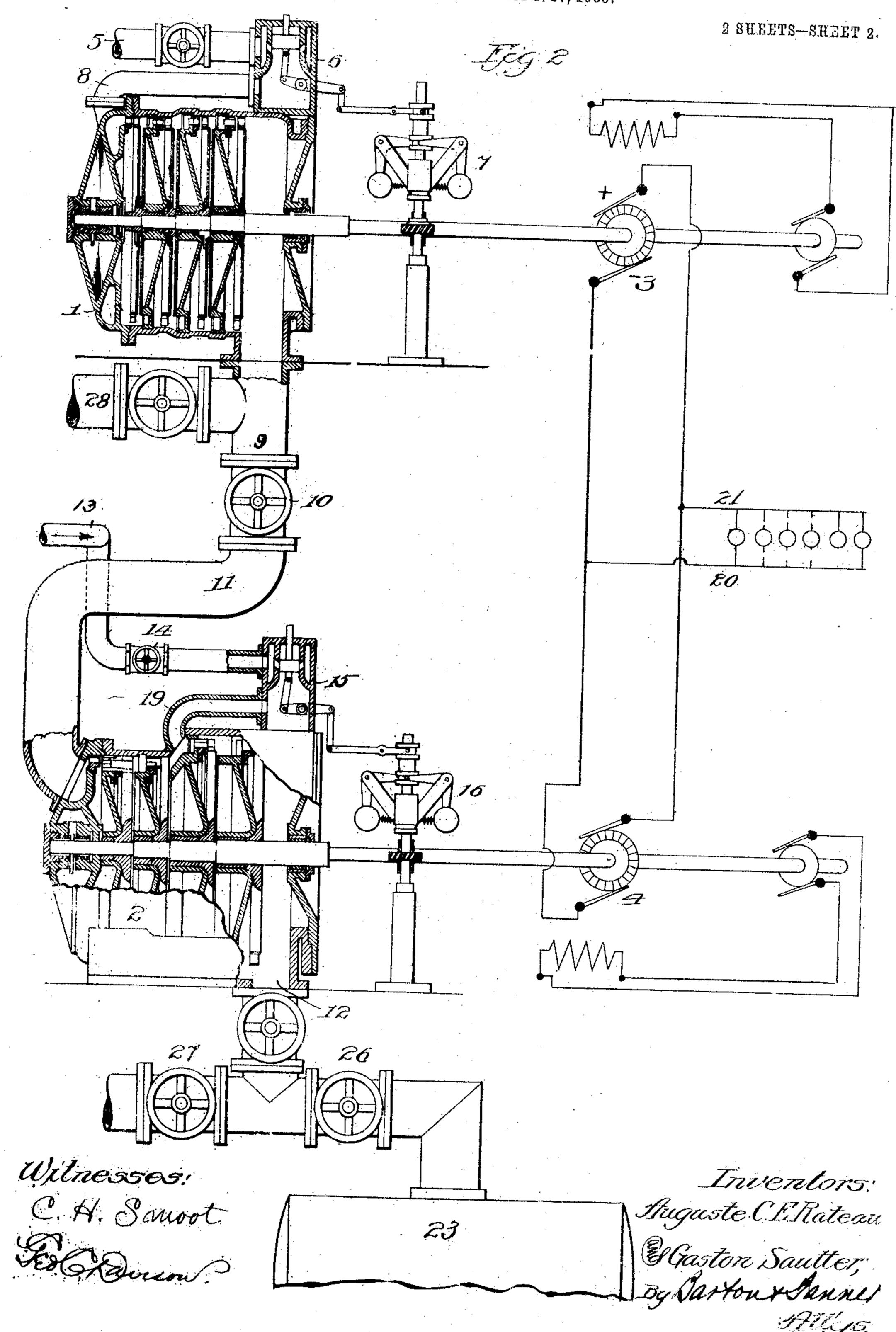
APPLICATION FILED AUG. 17, 1903



### PATENTED FEB. 20, 1906.

# A. C. E. RATEAU & G. SAUTTER. TURBINE DRIVEN MACHINERY.

APPLICATION FILED AUG. 17, 1903.



## UNITED STATES PATENT OFFICE.

AUGUSTE CAMILLE EDMOND RATEAU AND GASTON SAUTTER, OF PARIS, FRANCE, ASSIGNORS TO RATEAU TURBINE COMPANY, OF CHICAGO, ILLINOIS, A CORPORATION OF WEST VIRGINIA.

#### TURBINE-DRIVEN MACHINERY.

No. 812,878.

specification of Letters Patent.

Patented Feb. 20, 1906.

Application filed August 17, 1903. Serial No. 169,674.

To all whom it may concern:

Be it known that we, AUGUSTE CAMILLE EDof the Republic of France, residing at Paris, 5 France, have jointly invented certain new and useful Improvements in Turbine-Driven Machinery, of which the following is a specification.

This invention relates to the driving of electo tric generators or other power-transforming machines by turbines operated by steam, gas, or similar motive power; and its object is to provide an improved construction and arrangement whereby a single turbine unit of 15 very great power and speed may be employed and the total power developed thereby utilized most efficiently, as for the generation of electric currents.

The production of very heavy electric cur-20 rents by turbine-driven generators has heretofore been attended with great difficulties, because of the very high speed at which a turbine necessarily runs. While it is possible to construct such turbines so as to develop 25 enormous mechanical power, it is impractical to construct a generator of corresponding power which is capable of being direct driven by the turbine at the high speed which the turbine-shaft attains, such a speed being pro-30 hibitive for large generators.

In accordance with our invention the turbine-engine for developing the required total power is subdivided into high-pressure and low-pressure sections or individual turbines, 35 each section driving its own shaft, and a corresponding number of power - transforming machines, such as electric generators, are direct driven one by each of the several turbine - sections. The motive fluid is passed 40 through said individual turbines or sections in series to operate them together as a unit of motive power—that is to say, the motive fluid introduced into the first turbine of the series is permitted to expand and develop 45 mechanical power to a certain extent in said first turbine, after which the partially - expanded steam discharged from the first turbine is conveyed to the next turbine of the series, and so on until the total energy of the 50 steam is thus fractionally abstracted. The driven machines are connected in parallel with respect to the delivery of their output.

so as to react mutually and distribute the joint load between them in proportion to the MOND RATEAU and GASTON SAUTTER, citizens | power developed by their respective driving- 55 turbines.

A further feature of the invention consists in the provision of a single regulator mechanism, which acts in response to changes in the total load on the driven machines (as 6c manifested by changes in speed of both machines) to control the admission of steam or other motive fluid to the first turbine of the series.

Our invention further contemplates tem- 65 porarily increasing the power delivered by the series of driven machines by supplying additional live steam or motive fluid to a convenient intermediate point in the low-pressure section of the turbine. This enables the 70 driven machines to work on overload at certain times when the demand is greatest or to maintain the normal power of the combined turbines in case the condenser becomes inoperative and a discharge of the exhaust into 75 the open air becomes necessary.

With the general arrangement of turbines, driven machines, and regulating mechanism above described the individual driven machines may be of simple and strong construc- 80 tion and small in size, so as to be capable of operation at high speed, while efficiency is maintained by the practical absorption of the total power of the turbine, this being made possible by the division of said turbine 85 into high-pressure and low-pressure sections, each of which furnishes its proportionate

part of the total required power.

A further advantage of subdividing the turbine into two or more units lies in the fact 90 that it is sometimes advantageous to run the high-pressure turbine at a greater rotative speed than the low-pressure section. This is in consequence of the fact that the low-pressure section has a greater diameter than the 95 high-pressure section, and therefore cannot be rotated at so high a speed, because of excessive mechanical strains. By subdividing the turbine and letting each turbine drive a generator, the generators being connected in par- 100 allel, this may be readily accomplished, provided the generators are designed to operate in parallel at different rotative speeds.

We will describe our invention in detail by

reference to the accompanying drawings, in which—

Figure 1 is a diagrammatic representation of a turbine-driven electric generator set, con-5 structed, and arranged according to our plan, the generators being shown as alternatingcurrent machines. Fig. 2 is a diagram showing the parts more in detail and indicating the driven machines as direct-current gener-10 ators..

The same figures of reference designate the

same parts in both views.

As shown in the drawings, the turbine is divided into a high-pressure section 1 and a 15 low-pressure section 2, each of which drives its own independent shaft. A dynamo-electric generator 3 is direct-connected to the shaft of the high-pressure turbine 1, and a similar electric generator 4 is direct-connect-20 ed to the shaft of the low-pressure turbine. 2. In Fig. 1 the dynamos 3 and 4 are indicated as alternating-current machines, while in Fig. 2 said dynamos 3 and 4 are indicated as direct - current machines. In both cases the 25 terminals of said machines are connected in multiple with the mains 20 and 21 of the circuit to be supplied. Such connection acts to distribute the total electric load between the dynamos 3 and 4 in proportion to the 30 relative power developed by their respective driven turbines. The turbines are preferably constructed in accordance with our Patent No. 748,216, granted December 29, 1903, the present application being in sub-35 stance based upon the application, Serial No. 70,438, filed August 1, 1901, upon which said patent was granted. It is not essential to the present invention, however, that any particular type of turbine be employed, ex-40 cept that it should be divided into high-pressure and low-pressure sections, each section driving its own independent shaft. The steam arriving from the boilers by the pipe 5 passes into the first distributers of the high-45 pressure turbine through the pipe 8, an obturator 6 being provided to control the quantity of steam so admitted. A certain proportion of the energy of the steam is expended in driving the high-pressure turbine 1, and 50 from the exhaust-pipe 9 of said high-pressure turbine the steam is conveyed by way of pipe 11 to the low-pressure turbine 2. The steam, after finishing its work in the lowpressure turbine 2, leaves through the ex-55 haust-pipe 12 and may thence pass to a condenser 23. We have indicated in the drawings an outlet 24 into the open air for the steam from the exhaust 12 in case it is de-

sired for any reason to cut out the condenser.

outlet 28 for the steam from the exhaust-

pipe 9 in case it is desired to cut off the low-

pressure turbine altogether by closing the

valve 10. It will be understood, however,

6c We have also indicated in the drawings an

normally the exhaust-steam from the high pressure turbine passes through the pipe 11 into the low-pressure turbine, when it completes its work.

The compound turbine above described is 70 controlled during its normal operation by a single regulating mechanism, which may be associated with the high-pressure section 1. This mechanism comprises the obturator 6 and the speed-controlled regulating mechan- 75 ism 7 of the usual type for operating the same. This regulator mechanism is sensitive to slight changes of speed above or below the normal rate. Upon undue increase in speed it acts through the connecting-lever 80 mechanism to close the obturator 6, while if the speed falls below the predetermined rate said regulator acts to open the obturator and admit more steam. Variations of speed beyond the narrow limits determined by the 85 governing mechanism are thus prevented, the speed being maintained substantially constant.

We preferably provide means for temporarily augmenting the power of the turbine 90 when required to meet an extraordinarily heavy overload, such means consisting of a pipe 13, conveying live steam from the boiler to act on a section of the turbine which is intended for normal operation only at low 95 pressure. As shown in the drawings, such additional live steam may be introduced into an intermediate cell of the turbine 2, so as to act on the vane-wheels thereof toward the end of the series. The supply of such addi- 100 tional live steam may be controlled by a special obturator 15, which is governed by the regulating mechanism 16, the latter being shown as arranged to respond to changes in speed of the low-pressure turbine. This 105 special regulating mechanism is adjusted to act only in response to a fall of speed greatly below normal, as in case of an abnormally heavy overload, under which conditions the admission of additional live steam to the low- 110 pressure turbine will greatly increase its proportion of the total power delivered.

The operation of this apparatus is as follows: Under normal conditions the motive fluid is supplied to the high-pressure turbine 115 1, the extent of this supply being controlled by the obturator, which admits sufficient steam to maintain the speed of this turbine at approximately its normal value. The steam then passes through the turbine, de- 120 veloping a portion of the total work required, which energy is delivered in the form of electric power by the direct-connected dynamo. The steam on exhausting from turbine 1 passes into the low-pressure turbine 2, where 125 it completes its expansion, producing the rest of the available work contained in the steam, which work is converted into electric power by the direct-connected generator and 65 that this is only for an emergency and that | delivered to the system. On passing through 110

the second turbine the steam escapes through pipe 12 to the air or to the condenser, as may be required. The rotative speed of both units are maintained at approximately the 5 same value or at any predetermined approximately fixed ratio, dependent upon the design of the electric generators. The reason for this fixity of relative speeds is obvious in the case of an alternator, since the machines 10 operating in parallel are compelled to operate at synchronous speeds by virtue of the synchronizing current, which is exchanged between the two machines upon even a very slight departure from synchronous speed. In 15 the case of direct-current generators connected in parallel the same result follows, however, for a somewhat different reason. In this case if either machine attains a speed above that for which it generates normal 20 voltage its voltage will be increased accordingly, and will therefore exceed the voltage of the other machine whose speed has not been increased. In consequence a current will flow in series through the machines, said cur-25 rent acting to drive the slower-speed machine as a motor and being generated by absorbing power from the higher-speed generator. On account of this action it is impossible for either generator to depart mate-30 rially from the normal speed unless the other generator does so also. Consequently if the speed of either generator is controlled by a speed regulator or governor the speed of both generators is controlled. As a result of this 35 mutual relation of generators connected in parallel the two turbines perform their functions in much the same manner that they would operate were both connected to the same shaft and controlled by a single gov-40 ernor, as is the usual practice. Variations in the total load on the turbines will first cause an inverse variation in the rotative speeds of both sets, which speed variation actuates the governor on the first turbine of 45 the series and changes the supply of steam in accordance with the change in load. Under all conditions, however, the power delivered by each generator is determined by the power generated by the steam in the corresponding 50 turbine, except in cases where the speeds of the rotating elements are being accelerated or retarded. In this case some of the energy is supplied or absorbed by the acceleration of the rotating element. We have found that 55 with this arrangement the most suitable means of governing the speed of both turbines is obtained when the first turbine in series is equipped with a governor which admits sufficient steam to the first turbine to 60 maintain its speed at approximately the normal value for which the governor mechanism has been designed. The speed of the second or low-pressure unit is compelled to be approximately the same as that of the high-65 pressure unit by virtue of the electric genera-

tors of the two units being connected in parallel. It is obvious that since the speed of the two turbines must be practically the same one governor suffices to control the speed of both, although it actually controls 70 only the speed of the first turbine. Two turbines operating in this manner are in nearly the same condition as they would be were the two turbines mechanically coupled together by a slightly elastic coupling, said 75 coupling in the case of an alternator being elastic, but allowing no actual difference in the number of revolutions per minute of th two sets, while in the case of direct-current generators said coupling would be elastic and 80 would permit a small difference in the rotative speed of the two units. It is thus obvious that two units are arranged in this manner, controlled by a single governor actuated by the speed of the first turbine in the series, 85 insures that the steam-pressure distribution in the various sections of both turbines approximates very closely to the steam distribution which would be obtained were both turbines mechanically connected to the same go shaft. If each turbine were individually controlled by its own governor, said gov ernor acting to throttle the admission of steam into said turbine, the distribution of steam-pressures between the two turbines 95 will not be that which would be obtained with the two turbines connected to the same shaft, unless some complicated means were provided for distributing the electric load between the generators in such a way as to com- 100 pel this steam distribution. During normal operation the single regulating mechanism applied to the obturator 6 of the first turbine will suffice to keep the speed of the system approximately constant; but in case of an 105 overload upon the mains 20 21 so heavy that the full force of the steam admitted by the wide-open obturator 6 to the series of turbines is insufficient to furnish the required extra power the speed of the whole system 110 will fall considerably below normal. The regulator 16, controlling the obturator 15, is adjusted to respond to such abnormal fall of speed and will under such conditions act to open said obturator 15 and admit live steam 115 from pipe 13 to the turbine 2. There is thus added to the nearly-spent steam acting toward the end of the series of vane-wheels sufficient live steam to considerably increase the power developed by the turbine 2, and 120 thus to meet the additional demand upon the generators. During the continuation of the overload the regulator 16 serves, through variations in speed of the generator, to regulate, through the obturator 15, the admission 125 of live steam to the turbine according as the overload upon the dynamos 3 and 4 is increased or diminished. In the diagram Fig. 2 we have indicated the apparatus in the normal running condi- 130

tion, with the valve 6 open to admit steam to the first turbine of the series and the obturator 15 closed as it would be during the normal operation.

We claim—

1. The combination with a plurality of turbines connected in series with respect to the passage of steam, of a corresponding number of power-absorbing machines driven ro by said turbines, a connection between said driven machines to maintain approximately a fixed relative speed thereof, and a governor. mechanism responsive to changes in speed of the system controlling the admission of steam

5 to the first turbine of the series.

2. The combination with a plurality of turbines and means for conveying motive fluid through said turbines in series to operate the same as a unit of motive power, of a 20 corresponding number of power-transforming machines each driven by one of said turbines, said machines being connected in parallel with respect to the delivery of their output, and a single governor mechanism re-25 sponsive to changes in the speed of the system controlling the admission of motive fluid to the first turbine of the series.

3. The combination with two or more turbines and means for conveying motive fluid 30 through said turbines in sequence to operate the same as a unit of motive power, of a corresponding number of power-absorbing machines driven each by a corresponding one of said turbines, a regulator mechanism respon-35 sive to slight changes in speed of one of said turbines and arranged to control the admission of motive fluid to said series of turbines, and means for causing said driven machines to react mutually to distribute the total load 40 in proportion to the power developed by the

respective turbines.

4. The combination with high-pressure and low-pressure turbines, and means for. passing motive fluid through said turbines in 45 series to operate the same as a unit of motive power, of dynamo-electric generators driven each by a corresponding one of said turbines, and a main circuit to which said generators. are connected in parallel, whereby the load 50 upon said circuit at any instant is distributed between said generators in proportion to the power developed by their respective drivingturbines.

5. The combination with high-pressure 55 and low-pressure turbines, and means for passing motive fluid through said turbines in series, of electric generators driven each by a corresponding turbine, a main circuit fed with current from said generators, and con-60 nections between said generators adapted to distribute the total load of said main circuit between said generators proportionately to the power developed by their corresponding driving-turbines.

6. The combination with electric genera- 65 tors connected in parallel to a main circuit, of high-pressure and low-pressure turbines each driving a corresponding generator, means for causing motive fluid to pass through said turbines in series to operate the 70 same as a unit, an obturator controlling the supply of motive fluid to the first turbine of the series, and a governor for operating said obturator, responsive to changes in the total load on said main circuit.

7. The combination with a plurality of turbines connected in series with respect to the passage of motive fluid therethrough, of a corresponding number of power-absorbing machines driven by said turbines, a connec- 80 tion between said driven machines, means for admitting additional motive fluid to ar intermediate point in the turbine system, and a governor mechanism responsive to an abnormal fall of speed, arranged to control the 85

admission of such additional motive fluid. 8. The combination with a plurality of turbines connected in series, of a corresponding number of power-transforming machines driven by said turbines, a connection be- 90 tween said driven machines whereby they are caused to maintain approximately a fixed relative speed, a governor mechanism responsive to changes of speed of the system arranged to control the admission of motive 95 fluid to the first turbine of the series, means for admitting additional motive fluid to an intermediate point in the turbine system, and a second governor mechanism responsive to an abnormal fall of speed consequent 100 upon overload, arranged to control the admission of such additional motive fluid.

9. The combination with a plurality of turbines and means for passing steam through said turbines in series, of electric generators 105 driven one by each turbine, said generators being connected in parallel, a governor mechanism responsive to changes in speed of one of said turbines, controlling the passage of steam to the first turbine of the series, means 110 for supplying additional steam at an intermediate point in the turbine series, and a second governor mechanism responsive to an abnormal fall of speed consequent upon overload, arranged to control such additional 115 steam-supply.

. In testimony whereo' we have hereunto set our signatures in the presence of two sub-

scribing witnesses.

AUGUSTE CAMILLE EDMOND RATEAU. GASTON SAUTTER.

Witnesses:

C. DE MESTROL, Augustus E. Ingram.