

No. 784,225.

PATENTED MAR. 7, 1905.

O. S. PIKE.
PNEUMATIC DESPATCH APPARATUS.

APPLICATION FILED AUG. 6, 1903.

5 SHEETS—SHEET 1.

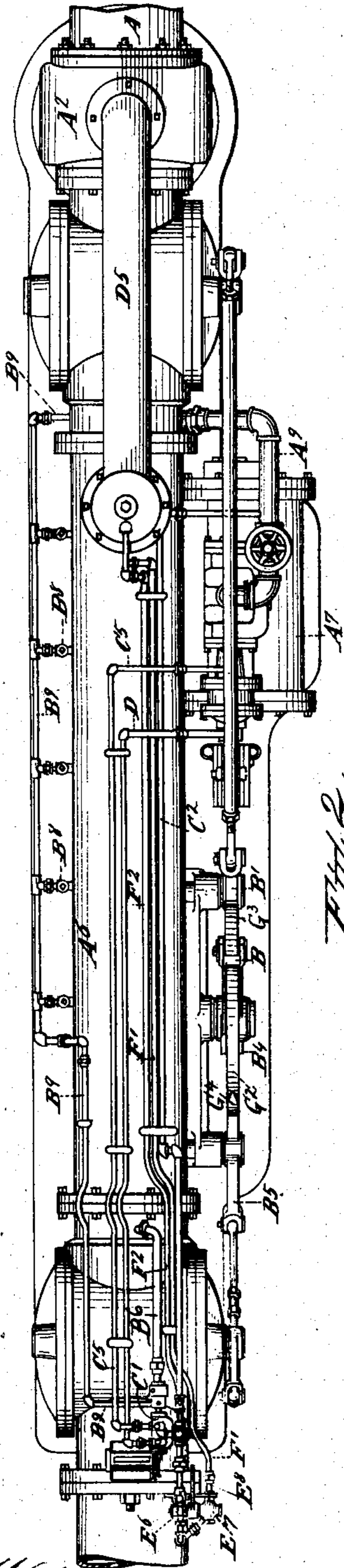


Fig. 2.

Witnesses:
A. R. Leland
Geo. Kemp

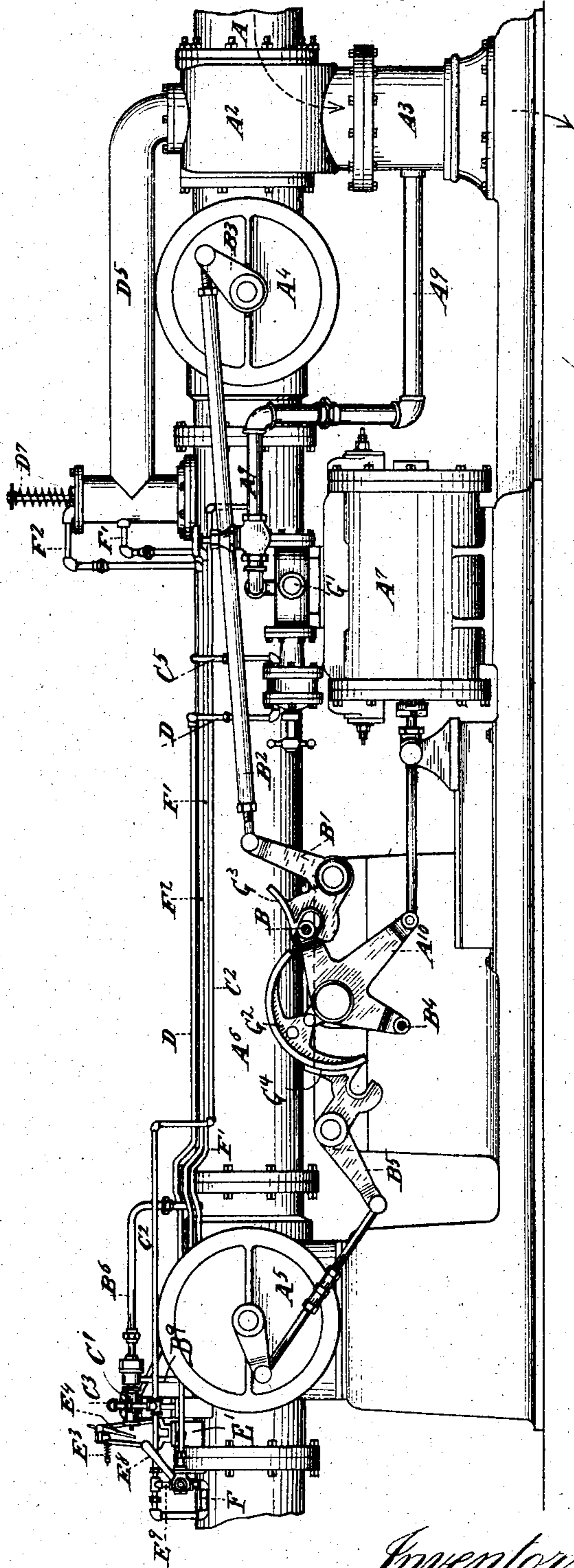


Fig. 1.

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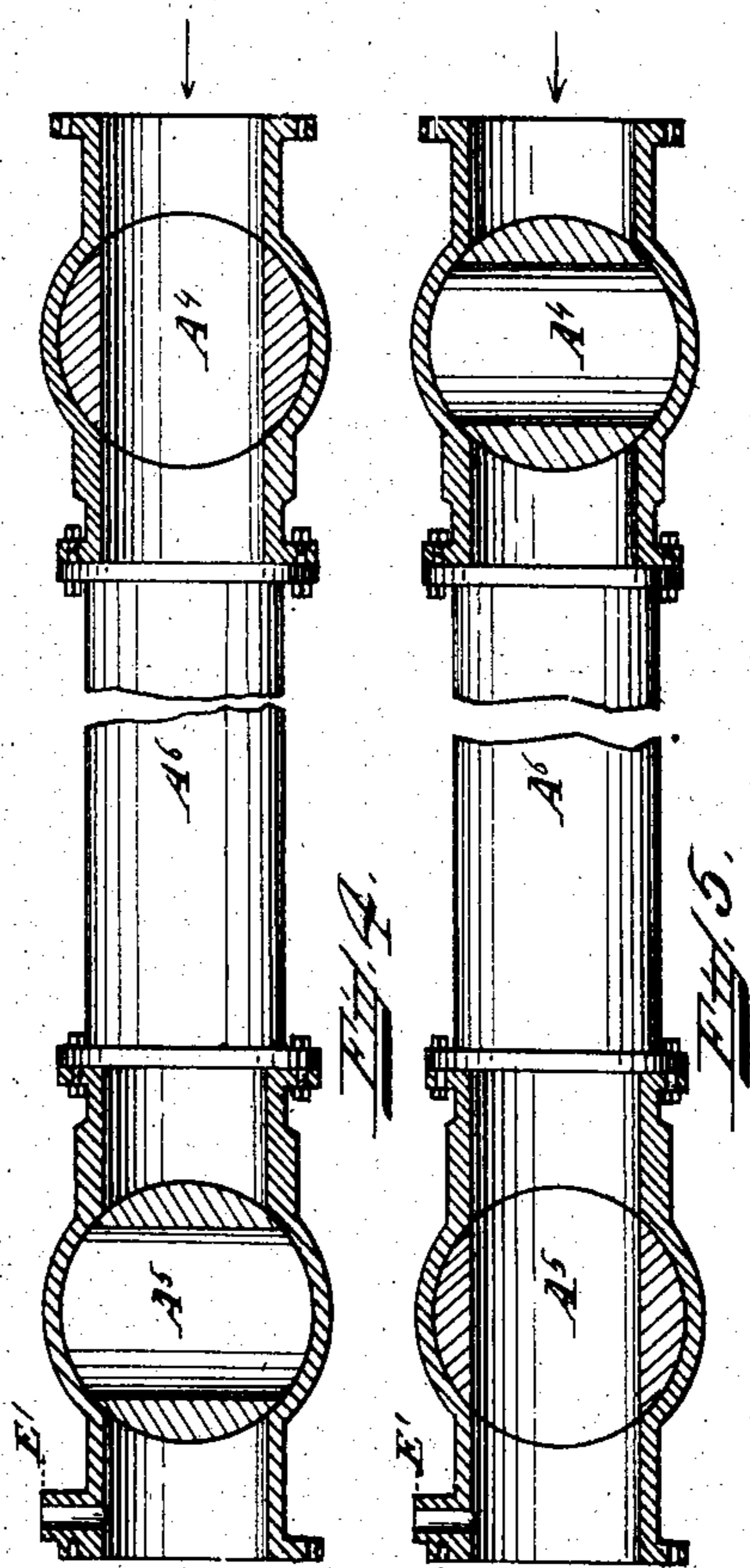
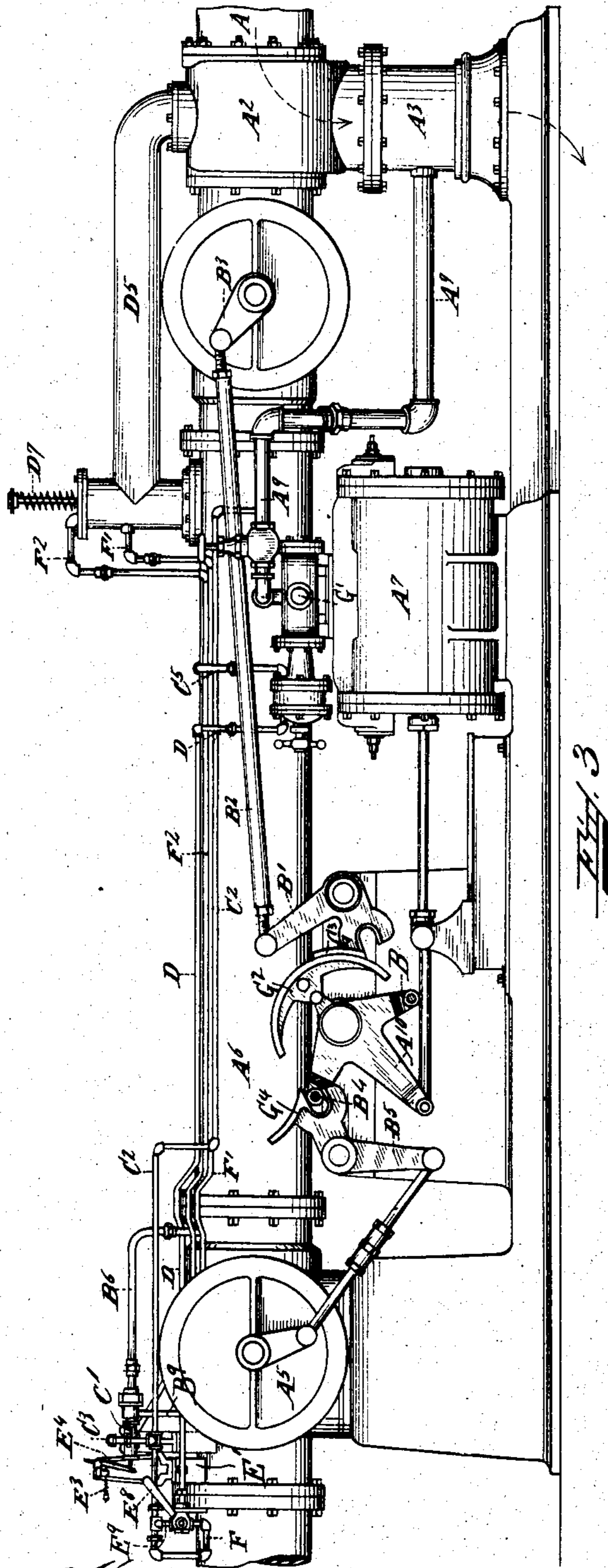
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5 SHEETS—SHEET 2.



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5 SHEETS—SHEET 3.

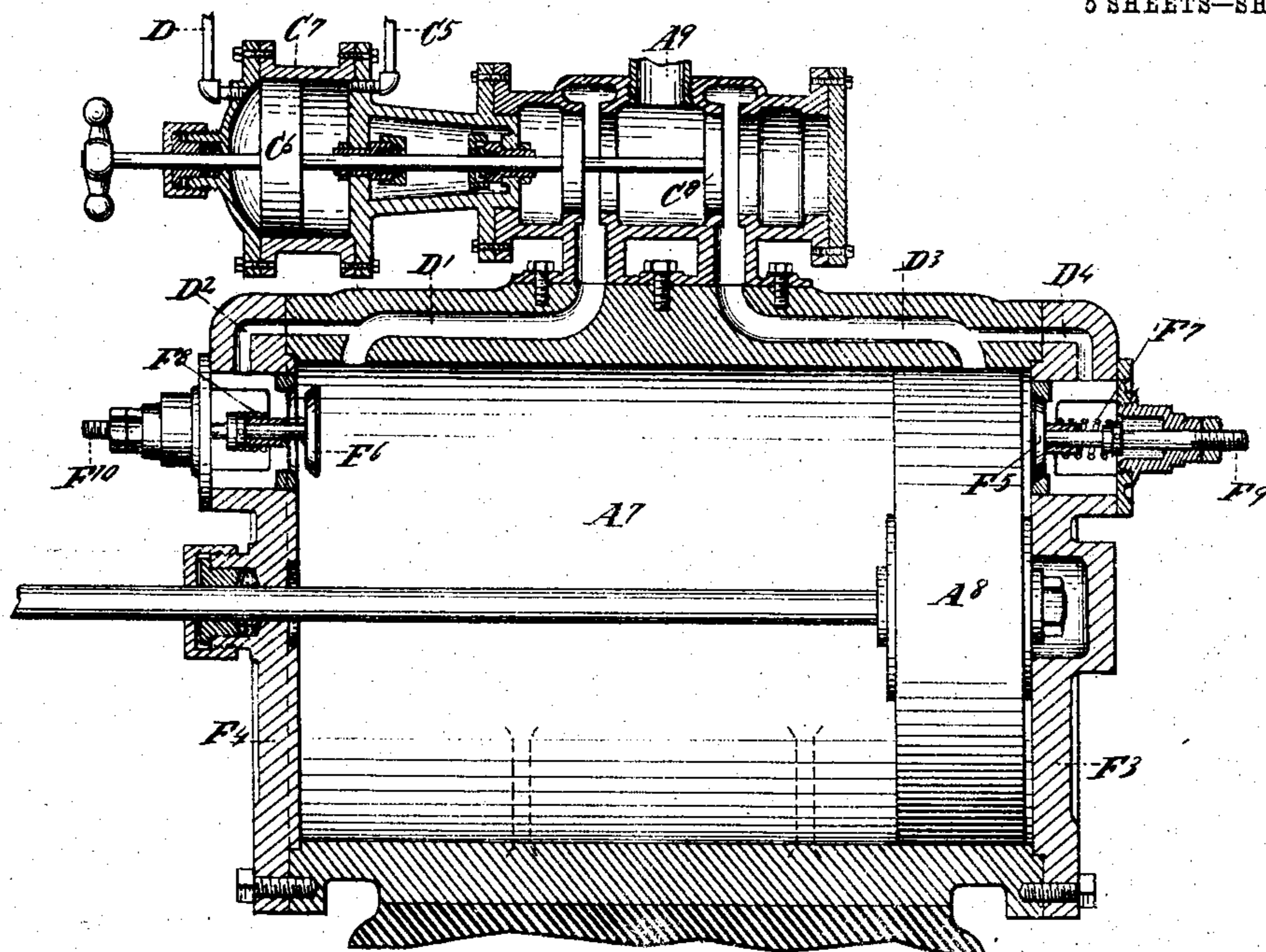


Fig. 6.

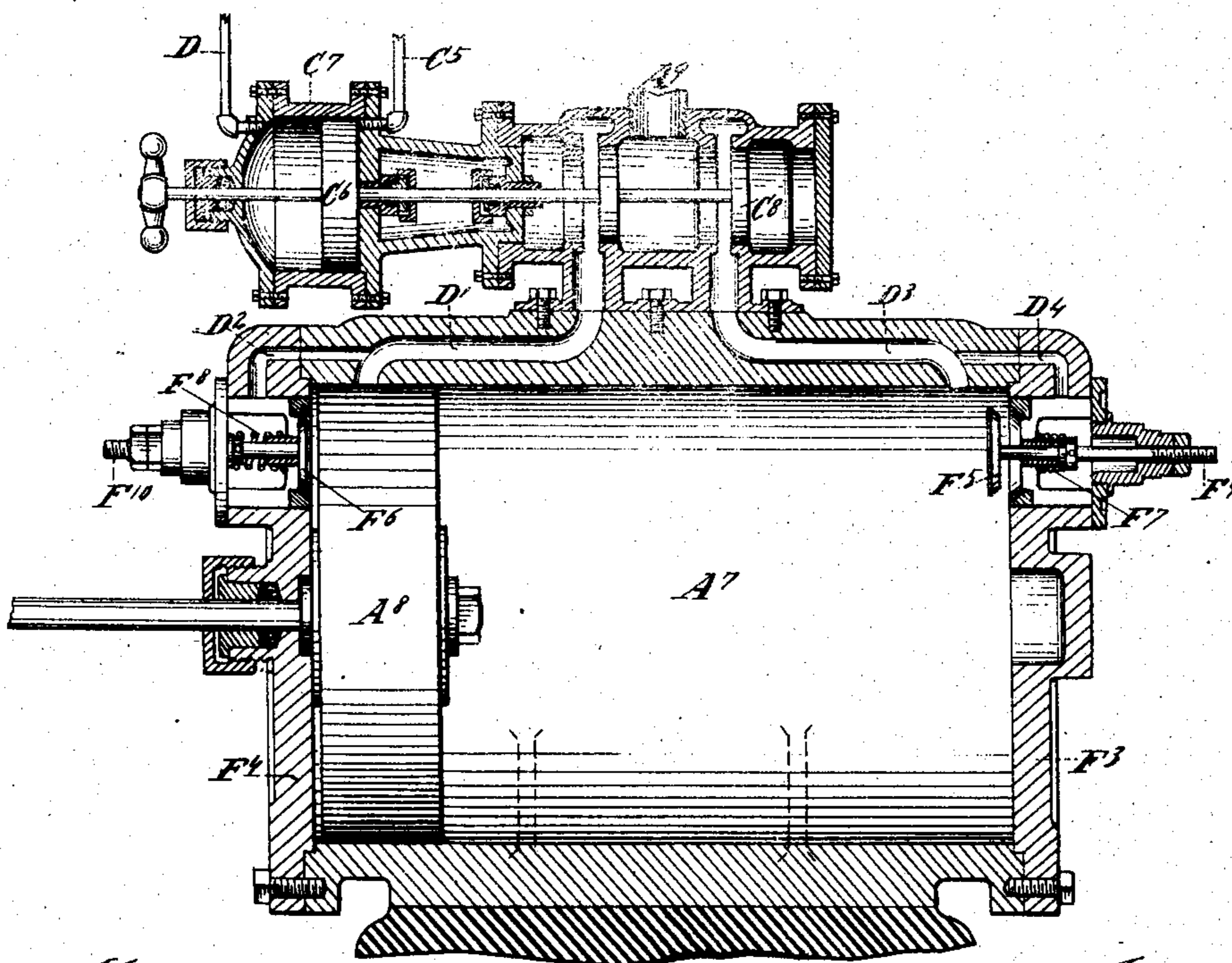


Fig. 7.

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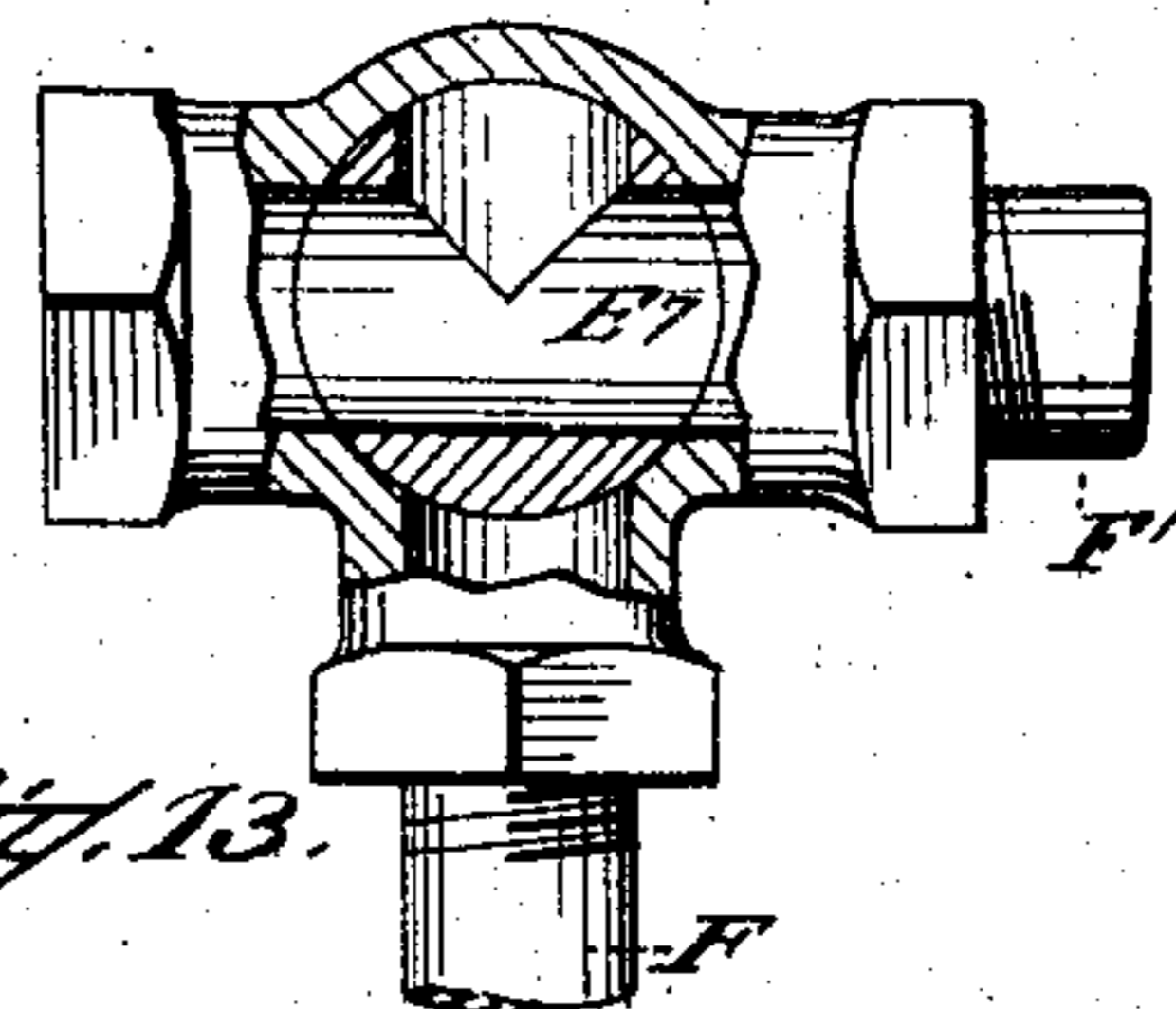
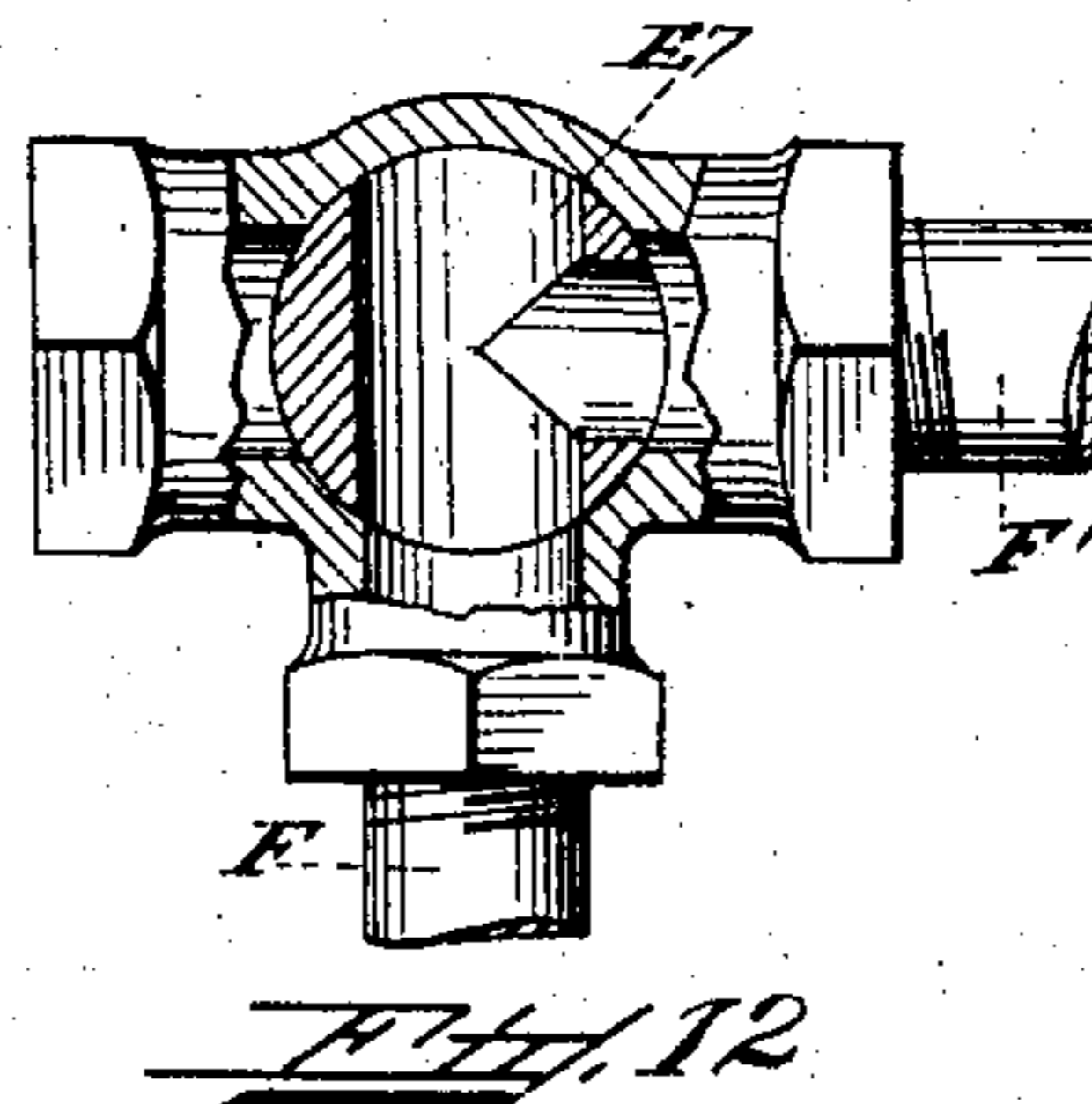
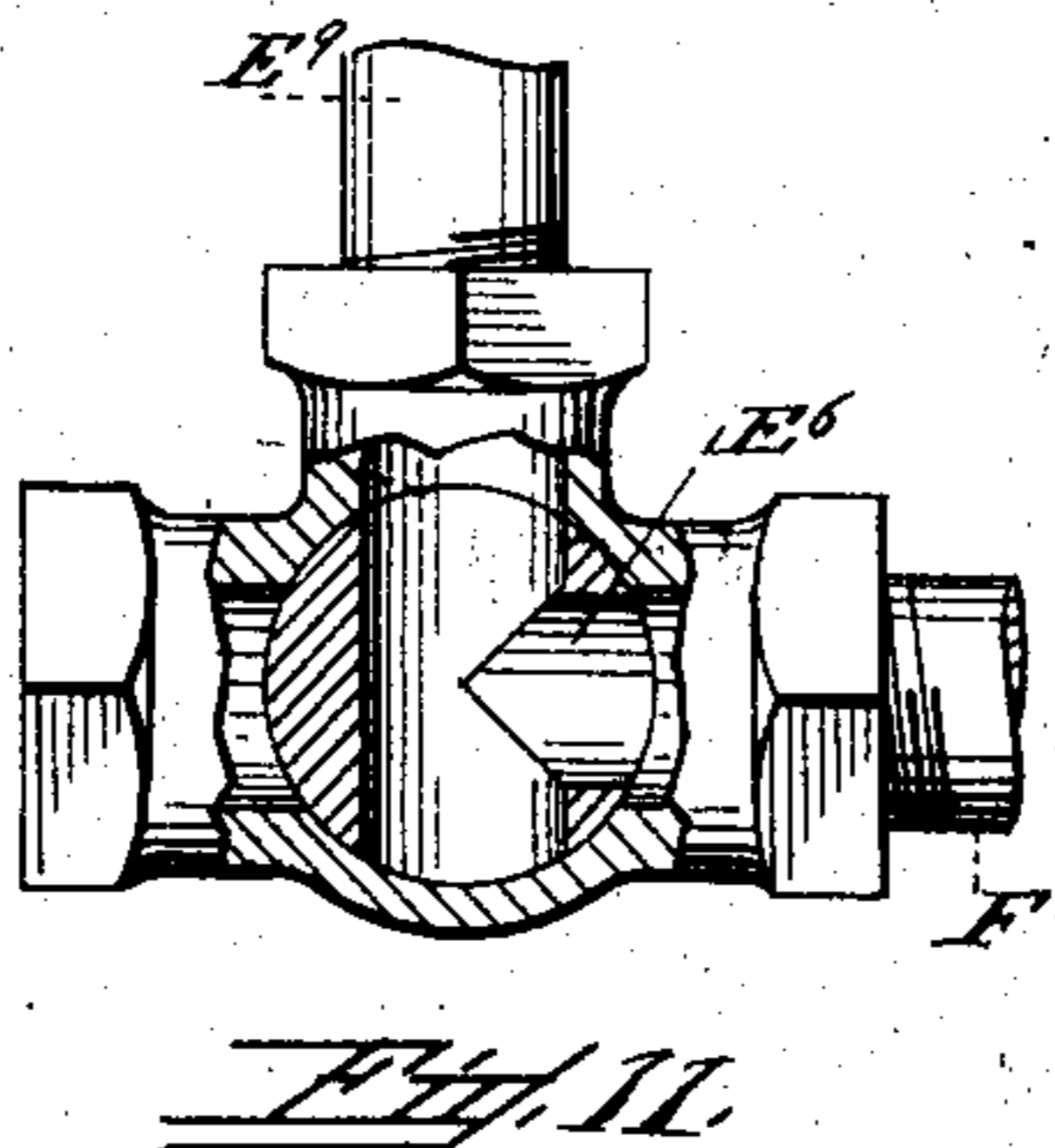
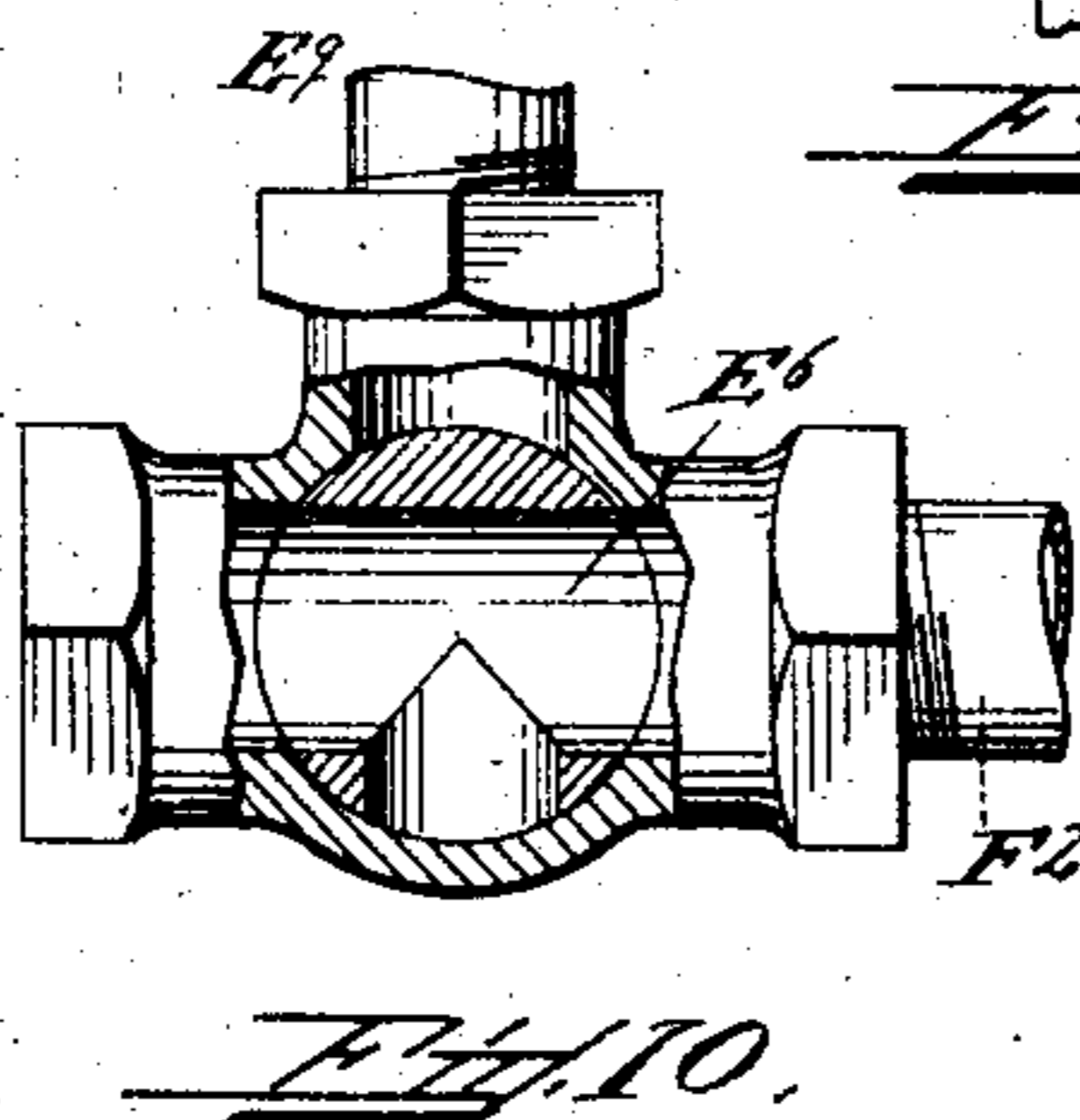
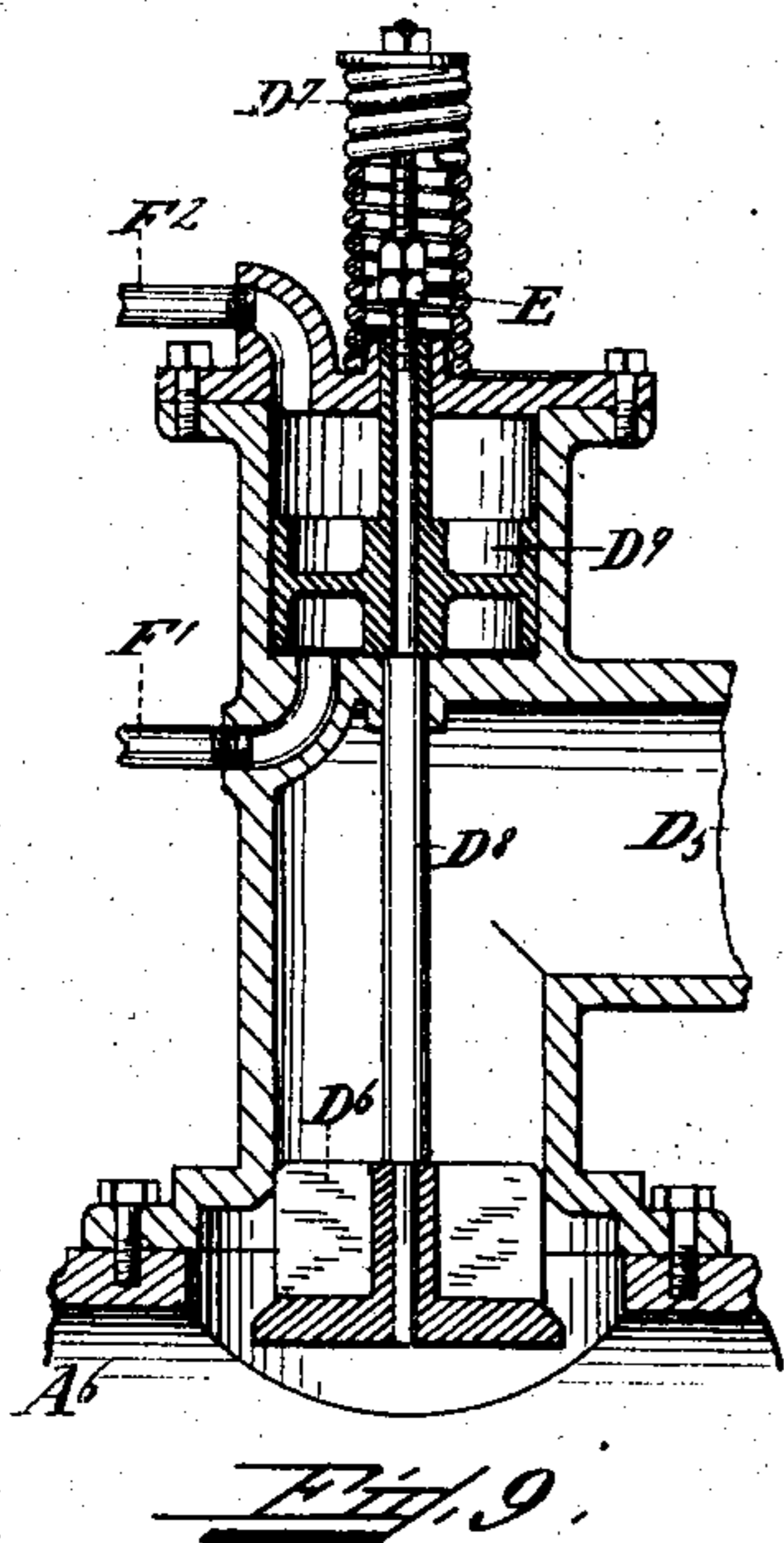
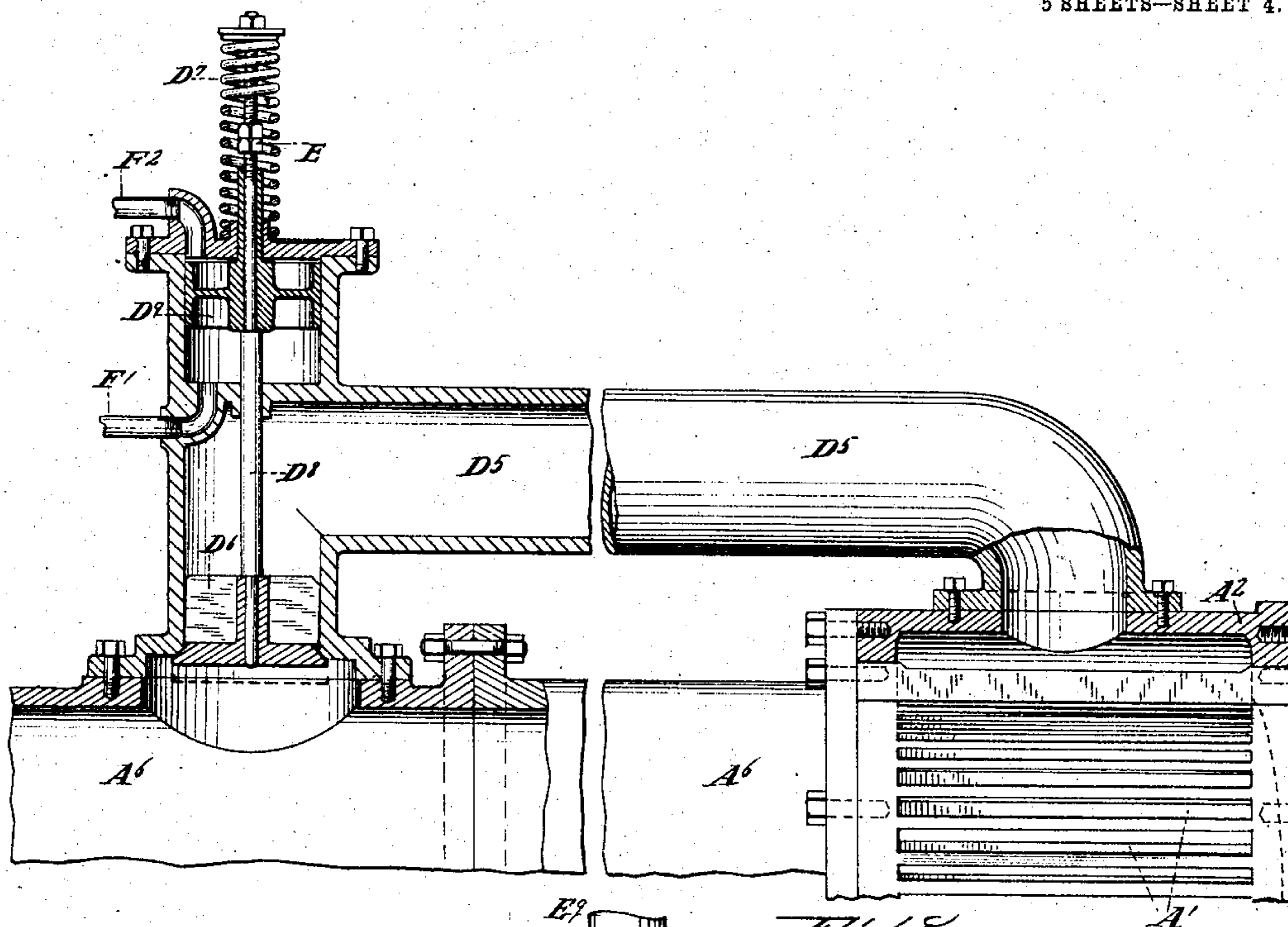
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5 SHEETS—SHEET 4.



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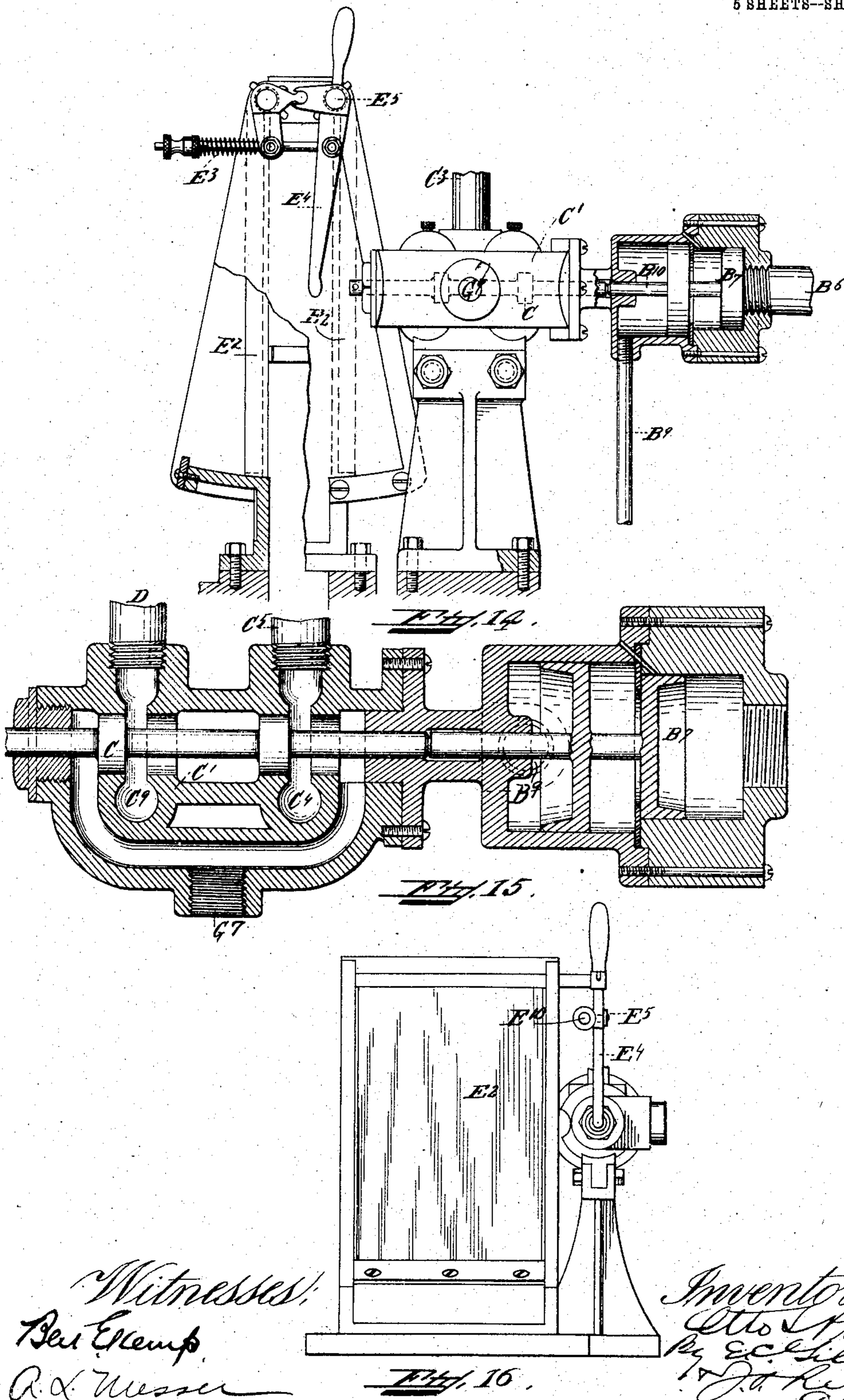
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5 SHEETS--SHEET 5.



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UNITED STATES PATENT OFFICE.

OTTO S. PIKE, OF MALDEN, MASSACHUSETTS, ASSIGNOR TO AMERICAN PNEUMATIC SERVICE COMPANY, OF DOVER, DELAWARE, A CORPORATION OF DELAWARE.

PNEUMATIC-DESPATCH APPARATUS.

SPECIFICATION forming part of Letters Patent No. 784,225, dated March 7, 1905.

Application filed August 6, 1903. Serial No. 168,468.

To all whom it may concern:

Be it known that I, OTTO S. PIKE, of Malden, in the county of Suffolk and State of Massachusetts, have invented certain new and useful Improvements in Pneumatic-Despatch Apparatus, of which the following is a specification.

My invention relates to new and useful improvements in closed receivers for pneumatic terminals.

The object of the invention is to remove carriers from a pneumatic-despatch tube while the tube is under pressure without causing a blast of air to flow from the tube into the atmosphere.

My invention consists, essentially, of a chamber to receive the carriers, a valve interposed between the chamber and the tube-line, the air from the tube-line being deflected before reaching said valve, another valve closing the above-mentioned chamber to the atmosphere. In action the carrier enters the chamber and the valve between the chamber and the tube-line closes. The valve between the chamber and the atmosphere opens, allowing the carrier to pass out onto a suitable table. Various auxiliary devices hereinafter described insure the prompt and correct working of the valves and the expulsion of the carrier from the terminal.

My invention consists of certain novel features hereinafter described, and particularly pointed out in the claims.

In the accompanying drawings, which illustrate a construction embodying my invention, Figure 1 shows an elevation of the terminal in its normal or closed position. Fig. 2 is a plan view of the same. Fig. 3 is an elevation of the terminal in its open position ready for a carrier to pass from the chamber onto the table. Fig. 4 is a section of the terminal through the valves, showing their position when the machine is closed. Fig. 5 is a similar section showing the position of the valves when the machine is open. Figs. 6 and 7 show a cylinder and piston with their attachments for operating the main valves in normal and open positions, respectively. Fig. 8 shows a by-pass pipe and valve in its normal position, the dotted lines showing the valve in open

position. Fig. 9 shows in section the by-pass valve and its piston in position for blowing out a carrier under certain circumstances, as hereinafter described. Figs. 10 to 16, inclusive, show various auxiliary portions of the machine hereinafter described.

Like letters of reference refer to like parts throughout the several views.

Referring to Figs. 1, 2, and 3, a portion of the tube-line is shown at A. The air travels through this tube in the direction indicated by the arrow and flows outward through the ports A', Fig. 8, into the T A² and away through the pipe A³. The valve A⁴ is now open and the valve A⁵ closed, as shown in Fig. 4. The carrier enters the T A², passing over the ports A', and travels through the valve A⁴ and into the receiving-chamber A⁶.

A⁷ is a cylinder in which is a piston A⁸, detail Figs. 6 and 7. A supply of compressed air from the pipe A³ or from any independent source is supplied to the cylinder A⁷ through the pipe A⁹. When air is admitted to the right of the piston A⁸, the piston is forced to the left and carries with it the wrist-plate A¹⁰. The roller B, mounted on the wrist-plate A¹⁰ and inclosed between the jaws of the bell-crank B', forces the bell-crank into position shown in Fig. 3. By means of the connecting-rod B² and crank B³ the valve A⁴ is rotated through a quarter-circle into the position shown in Fig. 5. When the valve A⁴ reaches this position, the roller B slips from the jaws of the bell-crank B', and the locking-arc G² engages with the curved portion G³ of the bell-crank B', thus holding the valve A⁴ securely in position. At about the same time the locking-arc G² disengages from the curved portion G⁴ of the bell-crank B⁵, and the roller B⁴, mounted on the wrist-plate A¹⁰, engages the jaws of the bell-crank B⁵, which operates the valve A⁵ and revolves the same into the position shown in Fig. 5.

The operation of the piston A⁸ is secured as follows: When a carrier enters the receiving-chamber A⁶, the air imprisoned in front of the carrier is compressed, thus bringing the carrier gradually to rest. The pipe B⁶

leads from the front end of the receiving-chamber to the right of the double piston B⁷. (Shown in detail in Figs. 14 and 15.) Fig. 14 shows the normal position of this piston.

5 At various points along the receiving-chamber A⁶ are tapped in small pipes B⁸. These pipes are supplied with stop-cocks, with the exception of the pipe nearest the valve A⁴. These pipes all connect to the pipe B⁹, which

10 leads to the left-hand side of the piston B⁷. The left-hand disk of the piston B⁷ is made larger than the right-hand disk, so as to offset the loss of area due to the piston-rod B¹⁰ and also insure the piston staying securely in

15 its normal position when no carrier is in the terminal. Any one or all of the stop-cocks in the pipes B⁸ may be left open. As a carrier enters the receiving-chamber A⁶ it passes by the pipes B⁸. When a carrier passes by

20 the nearest open pipe B⁸ to the valve A⁵, the excess of pressure in front of the carrier, due to cushioning of the carrier, acts only on the right hand of the piston B⁷, thus forcing said piston to the left, as shown in Fig. 15. When

25 the piston B⁷ moves to the left, it carries with it the piston-valve C in the valve-chest C'. A supply of compressed air passes from the pipe C² to pipe C³. Normally the air from the pipe C³ passes through the port C⁴ into

30 the pipe C⁵, thence to the left of the piston C⁶, which is in the cylinder C⁷, as shown in detail in Figs. 6 and 7. The piston C⁶ is attached to the piston slide-valve C⁸, the slide C⁸ controlling the supply of compressed air

35 for the piston A⁸ in the cylinder A⁷. When the valve C moves to the left, as noted above, the compressed air in the pipe C³ is led through the port C⁹ into the pipe D, thence to the left of the piston C⁶, forcing it and the piston slide-

40 valve C⁸ to the right. The air on the right of the piston C⁶ exhausts through the pipe C⁵ and port C⁴ through the opening G⁷ into the atmosphere. Normally the compressed air in the pipe A⁹ is led through the ports D'

45 and D² to the left of piston A⁸. When the piston-valve C⁸ is forced to the right, air is led to the right of the piston A⁸ through the ports D³ and D⁴, forcing the piston A⁸ to the left and rotating the valves A⁴ A⁵ successively,

50 as above noted, from the positions shown in Fig. 4 to those shown in Fig. 5, the air on the left of the piston A⁸ exhausting through the ports D' D² to the opening G' into the atmosphere. This cuts off the supply of air from

55 behind the carrier and opens the terminal in front of the carrier, the carrier being stationary, or nearly so, in the chamber A⁶. The by-pass pipe D⁵ (shown in detail in Fig. 8) leads from the T A² to the receiving-chamber A⁶ at

60 a point just in front of the valve A⁴. In the pipe D⁵ is a valve D⁶, which is held normally closed by the spring D⁷ and by the excess air-pressure on the bottom of the valve D⁶, the pressure on the bottom being somewhat

65 greater than that on top, owing to the conical

form of the valve. The valve-spindle D⁸ slides freely in the piston D⁹. When the valves A⁴ and A⁵ move into the positions shown in Fig. 5 after the arrival of a carrier in the chamber A⁶, the air-pressure underneath the

70 valve D⁶ is dissipated into the atmosphere, leaking by the carrier or pushing the carrier outward. This allows the air-pressure on top of the valve D⁶ to force said valve open against

75 the spring D⁷ into the position shown in broken lines in Fig. 8, the amount of opening being regulated by the nut E on the valve-spindle D⁸. The opening of the valve D⁶ allows sufficient compressed air from the main

80 tube-line to pass through the pipe D⁵ into the chamber A⁶ behind the carrier, and thus force the carrier out onto the table promptly. When the valve D⁶ moves downward, the piston D⁹ remains stationary. The use of this piston

85 will be described later. As the carrier passes from the receiving-chamber A⁶ onto the receiving-table part of the air behind it is deflected upward through the port E', Figs. 4 and 5,

90 which is just in front of the valve A⁵. The air then passes between the vanes E². (Shown in detail in Figs. 14 and 16.) These vanes are normally held in the position shown in Fig. 14 by the spring E³. The air-pressure between these vanes forces them outward, and

95 by means of the lever E⁴, which is rigidly attached to the spindle E⁵ of one of the vanes, the valve C is forced to the right, carrying with it the piston B⁷. When the valve C moves

100 to the right, air from the pipe C³ passes through the port C⁴ into the pipe C⁵, thence to the right of the piston C⁶, forcing this piston and the piston-valve C⁸ into the position

105 shown in Fig. 6, thus admitting air to the left of the piston A⁸ and bringing the valves A⁵ A⁴ successively into their normal positions, as shown in Fig. 4. When the carrier has passed

110 out of the terminal, the pressure behind the vanes E² dissipates and allows the vanes to assume their normal position. When the valve A⁴ opens the main tube-line to the chamber

115 A⁶, the pressure in the receiving-chamber A⁶ becomes the same as that in the pipe D⁵, so that the spring D⁷ may close the valve D⁶. This restores the terminal to its normal condition.

In Figs. 10 and 11 are shown sections of a three-way cock E⁶, Fig. 10 showing the normal position. Figs. 12 and 13 show similar sections of another three-way cock, E⁷, Fig. 12 showing the normal position. These cocks

120 E⁶ E⁷ are manually operated simultaneously by the lever E⁸. Compressed air is supplied through the pipe E⁹ to the cock E⁶ and through the pipe F to the cock E⁷. The pipes E⁹ and F lead from the pipe C². Normally the pipe

125 E⁹ is closed by the cock E⁶ while air from the pipe F passes through the cock E⁷ and the pipe F' to the bottom of the piston D⁹ above the by-pass pipe D⁵, holding the piston in its

130 upper position, as shown in Fig. 8, the air

above the piston being exhausted through the pipe F² and valve E⁶ into the atmosphere. In case a carrier sticks so hard in the receiving-chamber A⁶ (owing to excessive friction, dirt, or other causes) that the air which passes by the valve D⁶ is insufficient to force the carrier out, the attendant throws the lever E⁸ to the left, thus rotating the cocks E⁶ and E⁷ into the positions shown in Figs. 11 and 13, respectively. This exhausts the air beneath the piston D⁹ into the atmosphere through the pipe F⁷ and the cock E⁷ and admits the compressed air through the pipe E⁹ and valve E⁶ to the top of the piston D⁹, thus forcing the piston D⁹, and with it the valve D⁶, to its extreme downward position, as shown in Fig. 9, and admitting a larger supply of air through the by-pass pipe D⁵ into the receiving-chamber A⁶ behind the carrier and forcing the carrier out through the valve A⁵ onto the receiving-table. After the carrier has passed out of the machine, the lever E⁸ may be turned to its normal position, as shown in Fig. 1, thus restoring the piston D⁹ to its normal position, the spring D⁷ drawing the valve D⁶ upward.

Referring to Figs. 6 and 7, in the cylinder-heads F³ F⁴ are the check-valves F⁵ F⁶. These valves open inwardly, and the springs F⁷ and F⁸ tend to hold them in the closed position. Normally both valves F⁵ F⁶ would be closed on their seats unless prevented by the adjusting-screws F⁹ F¹⁰, hereinafter described. In Fig. 6, however, the valve F⁶ is shown in the position it would take when the piston A⁸ is moving to the right. The purpose of this valve is to secure an air-cushion on the end of the cylinder to prevent the piston A⁸, and consequently the valves A⁴ A⁵, coming home with a shock. As the piston A⁸ moves to the right it overlaps the opening of the port D³, and the pressure generated in the closed space, combined with the spring F⁷, tends to keep the valve tightly closed. The valve F⁵, however, may be kept from entirely closing by the adjusting-screw F⁹, so that the air may escape by the valve F⁵ slowly and allow the piston A⁸ to come home slowly. When air is admitted to the port D³, so as to force the piston A⁸ to the left, some of the air can pass through the port D⁴ to the right of the valve F⁵ and by forcing the valve F⁵ open can pass into the cylinder to the right of the piston A⁸. After the piston A⁸ has moved a short distance the opening of the port D³ is uncovered, thus allowing the full volume of air to enter the cylinder. The action of the valve F⁶ is entirely similar to the valve F⁵, previously described.

Having thus described the nature of my invention and set forth a construction embodying the same, what I claim as new, and desire to secure by Letters Patent of the United States, is—

1. In an apparatus of the character described, a terminal, a transmission-tube, a

valve for closing said terminal to the atmosphere and normally closed, a valve for closing communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply to said cylinder for operating said piston, an air-supply for operating said cylinder-controlling valve, and a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the traveling carrier for moving said piston in the opposite direction.

2. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, a locking device for holding said valves in the position to which they are moved by the operation of the piston, an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply to said cylinder for operating said piston, an air-supply for operating said cylinder-controlling valve, and a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the traveling carrier for moving said piston in the opposite direction.

3. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, a locking device for holding said valves in the position to which they are moved by the operation of the piston, an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply to said cylinder for operating said piston, an air-supply for operating said cylinder-controlling valve, a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the traveling carrier for moving said piston in the opposite direction, and

mechanism operated by the pressure at the rear of the traveling carrier for operating said valve-operating mechanism to return said valves to their normal positions.

5 4. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing communication between said terminal and the
10 transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, a locking device for holding said valves in the position to which
15 they are moved by the operation of the piston, an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply for operating said cylinder-
20 controlling valve, a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the
25 traveling carrier for moving said piston in the opposite direction, and mechanism consisting of two vanes and operated by the pressure at the rear of the traveling carrier to operate said valve-operating mechanism to return said
30 valves to their normal positions.

5 5. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing
35 communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, an air-supply for
40 operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply to said cylinder for operating said piston, an air-supply for operating said cylinder-controlling
45 valve, and a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the traveling car-
50 rier for moving said piston in the opposite direction, and mechanism operated by the pressure at the rear of the traveling carrier for operating said valve-operating mechanism to return said valves to their normal positions.

55 6. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing communication between said terminal and the
60 transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, a locking device for holding said valves in the position to which
65 they are moved by the operation of the piston,

an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply for operating said cylinder-controlling valve, a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the traveling carrier for moving said piston in the
7 opposite direction, mechanism consisting of two vanes and operated by the pressure at the rear of the traveling carrier to operate said valve-operated mechanism to return said
8 valves to their normal positions, and means for normally holding said vanes against movement.

7. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing said terminal to the atmosphere and normally closed, a valve for closing
9 communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a cylinder, a piston in said cylinder connected to said mechanism, a locking device for holding said valves in the position to which
10 they are moved by the operation of the piston, an air-supply for operating said piston to move said valves, a valve operated by compressed air for controlling the flow of air from said air-supply for operating said cylinder-
11 controlling valve, a valve controlling the air-supply which operates said cylinder-controlling valve and operated by the pressure in the terminal for moving the piston in one direction and actuated by air compressed by the
12 traveling carrier for moving said piston in the opposite direction, mechanism consisting of two vanes and operated by the pressure at the rear of the traveling carrier to operate said
13 valve-operating mechanism to return said valves to their normal positions, and yielding means for normally holding said vanes against movement.

8. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing communication between said terminal and the transmission-tube, a by-pass from said transmission-tube to said terminal, a valve normally closing said by-pass, means for holding said valve closed, a cylinder, a piston in said cylinder for operating said valve, a supply of compressed air for operating said piston, and means for controlling said
14 air-supply to move said piston in opposite directions.

9. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing communication between said terminal and the transmission-tube, a by-pass from said transmission-tube to said terminal, a valve normally closing said by-pass, means for holding said valve closed, a cylinder, a piston in said cylinder for operating said

valve, a supply of compressed air for operating said piston, pipes for the compressed air leading to opposite sides of said piston, and means for controlling the supply of compressed air through said pipes to move the piston in opposite directions.

10. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing communication between said terminal and the transmission-tube, a by-pass from said transmission-tube to said terminal, a valve normally closing said by-pass, means for holding said valve closed, a cylinder, a piston in said cylinder for operating said valve, a supply of compressed air for operating said piston, pipes for the compressed air leading to opposite sides of said piston, and manually-operated valves for controlling the flow of compressed air through said pipes to move the piston in opposite directions.

11. In an apparatus of the character described, a terminal, a transmission-tube, a valve for closing communication between said

terminal and the atmosphere and normally closed, a valve for closing communication between said terminal and the transmission-tube and normally open, mechanism common to said valves for operating the same, a by-pass from said transmission-tube to said terminal, a valve normally closing said by-pass, and means for holding said by-pass valve normally closed, the said by-pass valve being opened by the pressure in the transmission-tube through said by-pass when the valve between the terminal and the transmission-tube is closed and the valve between the terminal and the atmosphere is opened.

In testimony whereof I have signed my name to this specification, in the presence of two subscribing witnesses, this 30th day of July, A. D. 1903.

OTTO S. PIKE.

Witnesses:

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E. L. HARLOW.