No. 689,738.

Patented Dec. 24, 1901.

## E. MORIARTY.

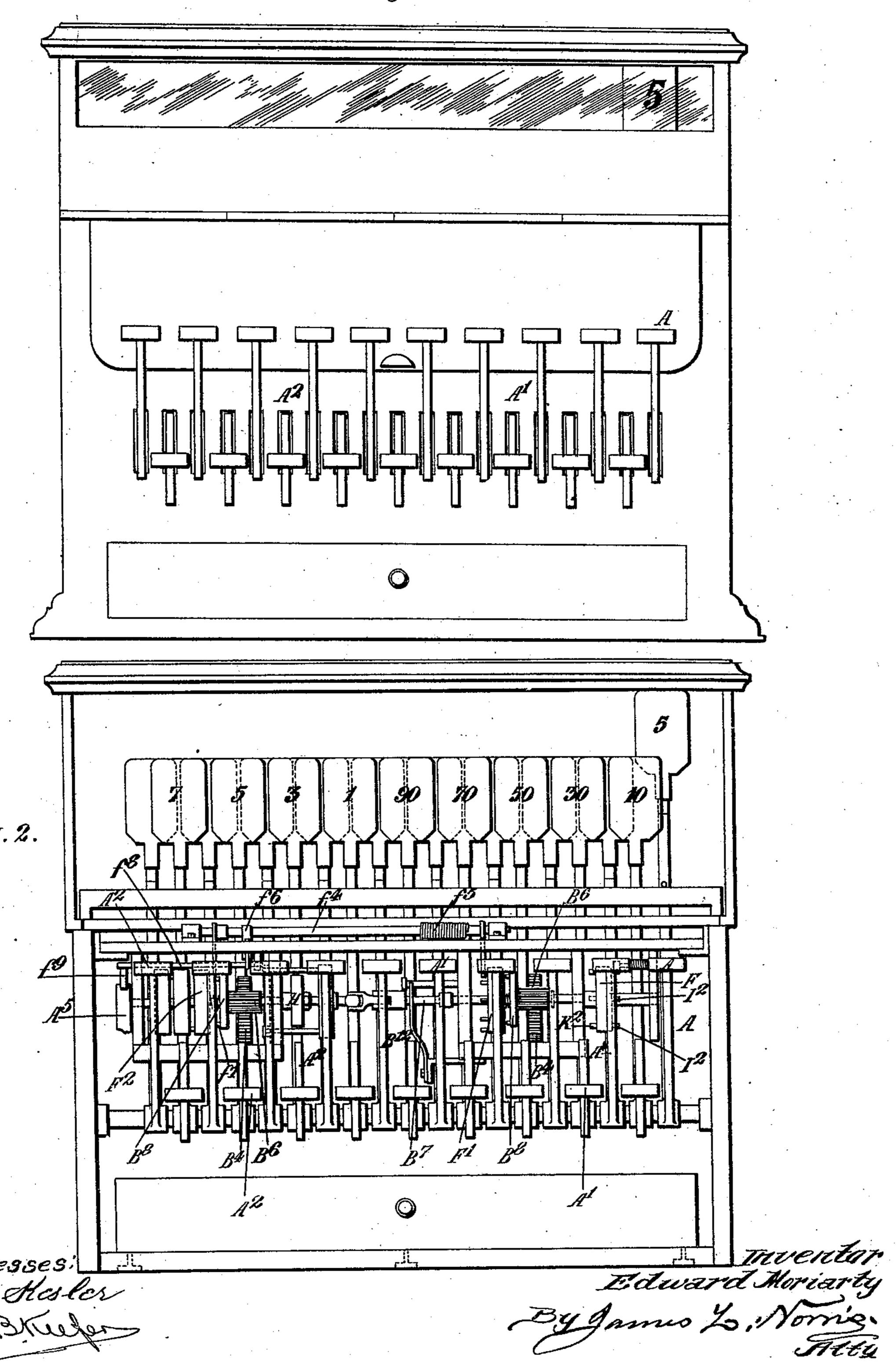
#### COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

(No Model.)

7 Sheets—Sheet I.

Fig.1.

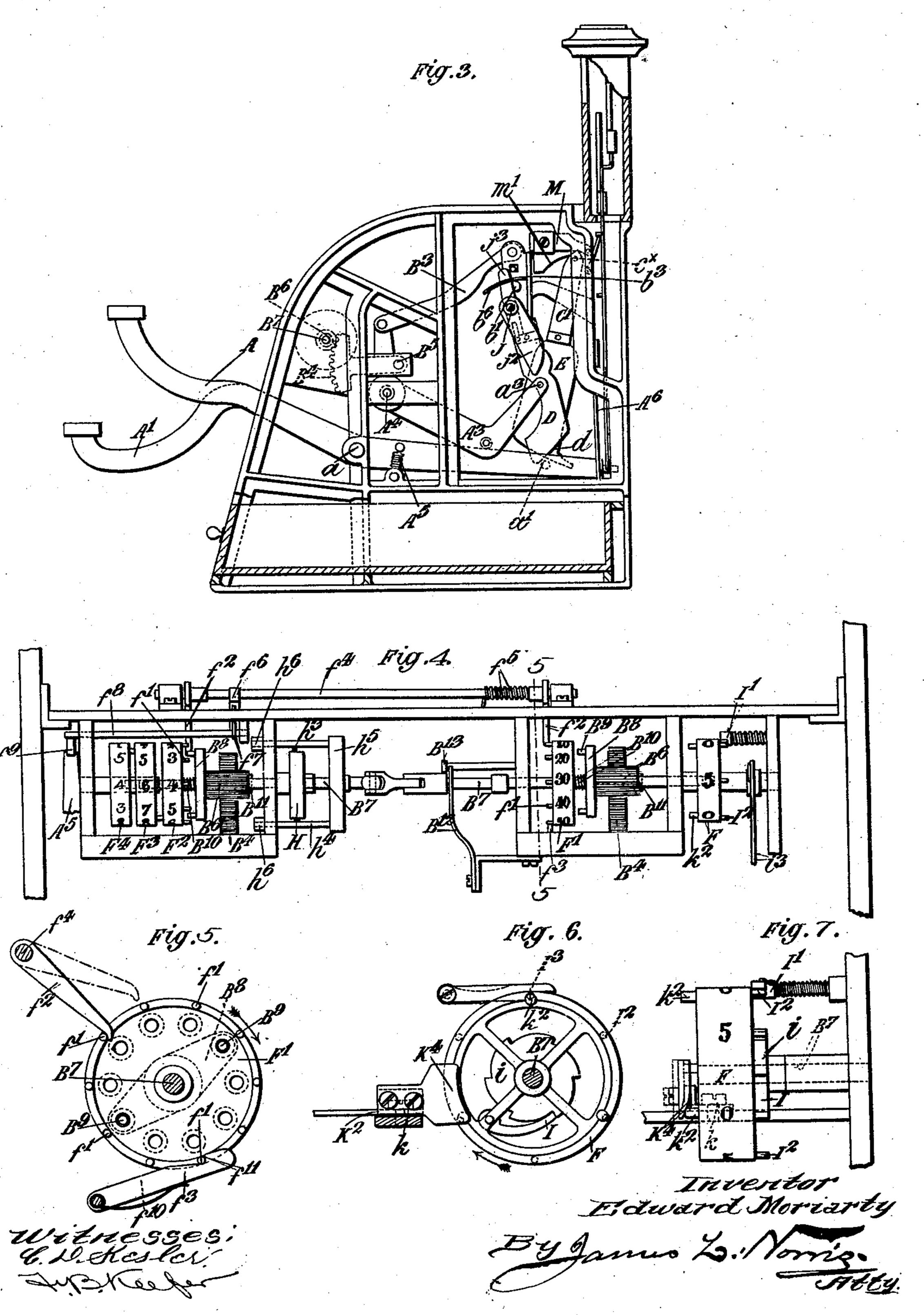


#### COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

(No Model.)

7 Sheets—Sheet 2.

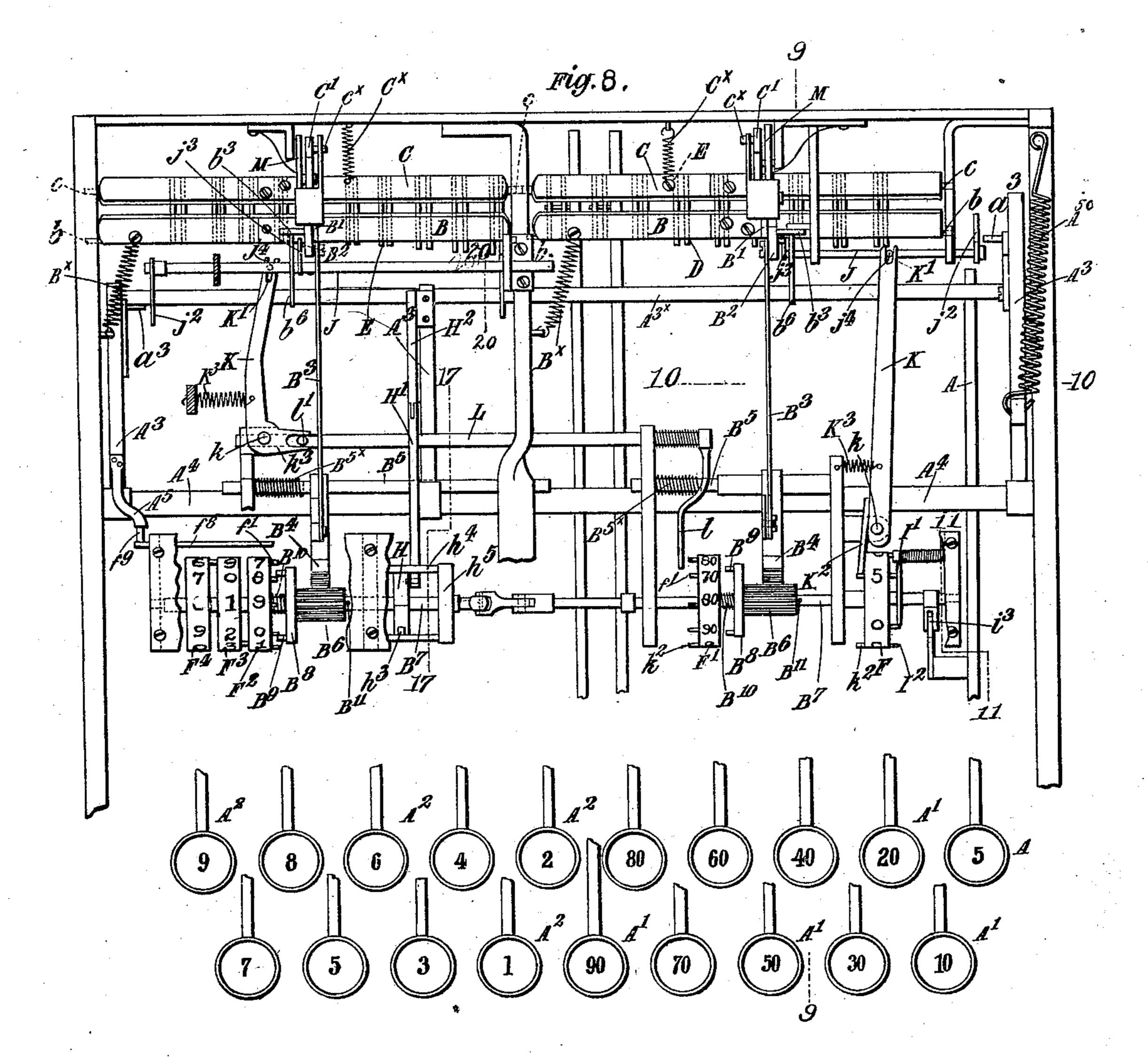


#### COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

(No Model.)

7 Sheets—Sheet 3.



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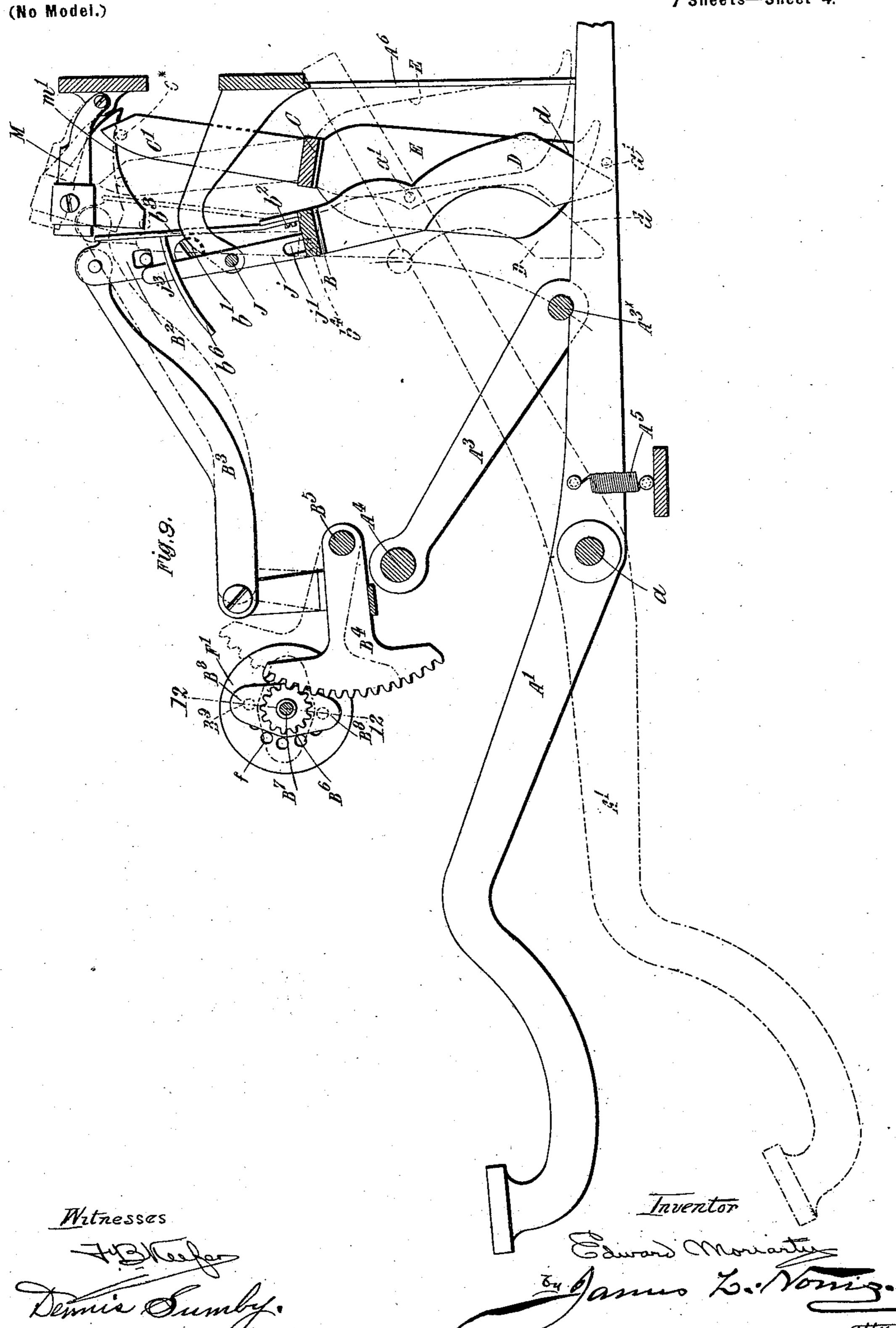
Edward Moriarty

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# COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

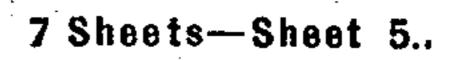
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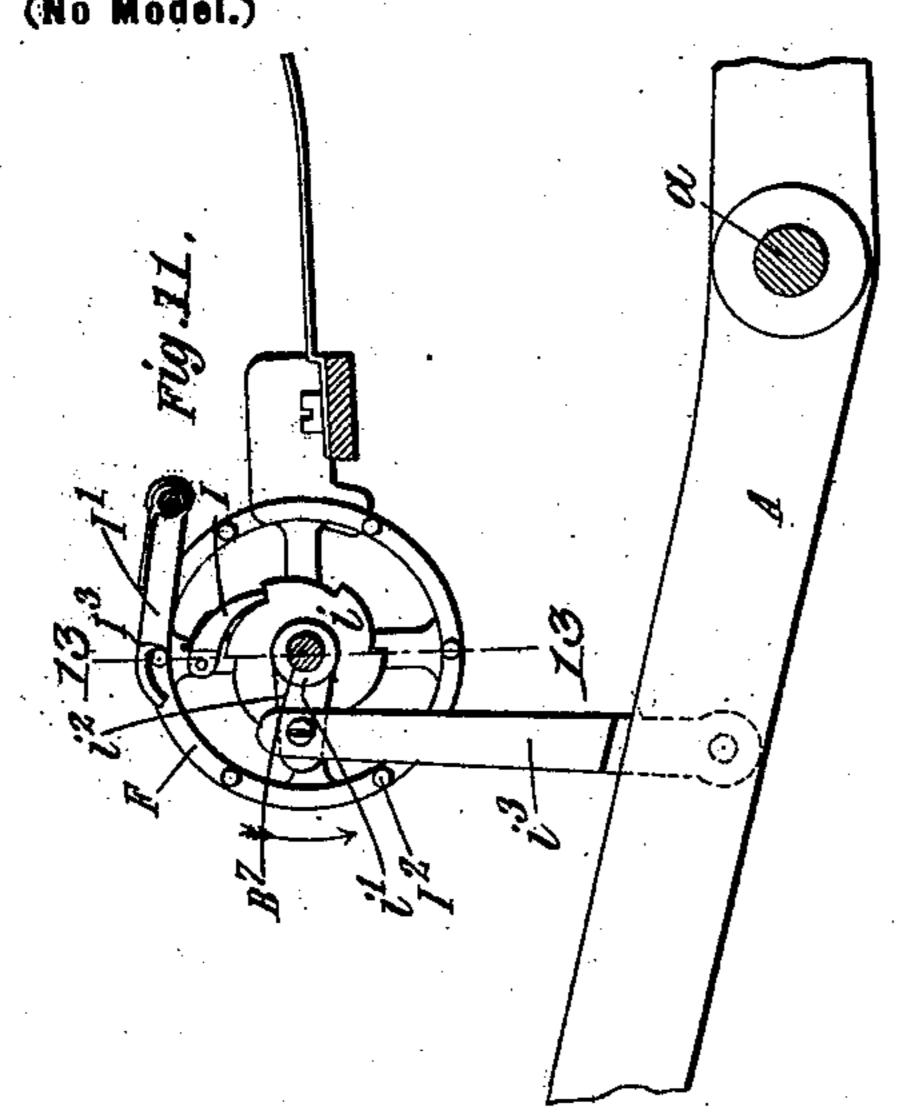


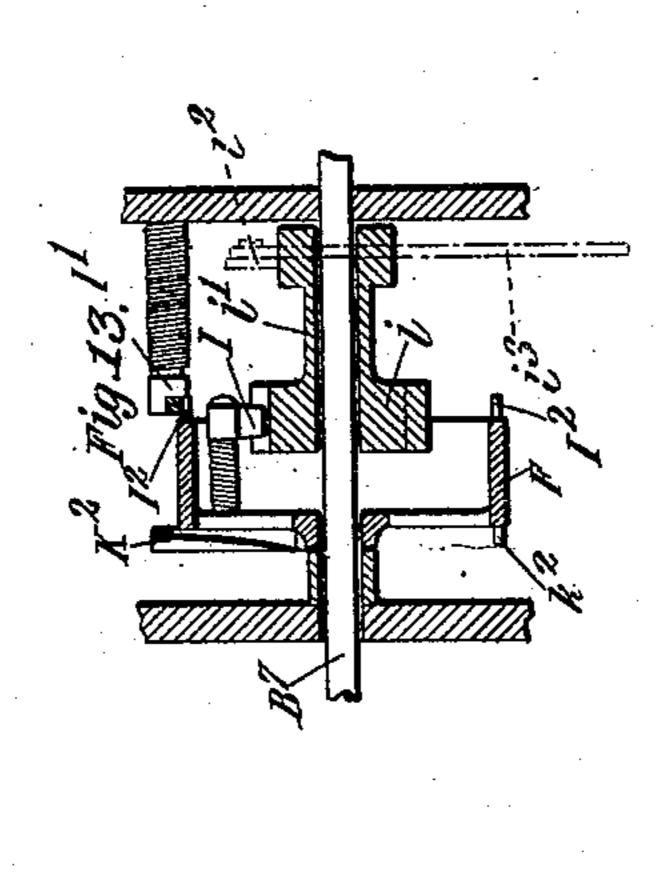
#### COUNTING OR ADDING APPARATUS.

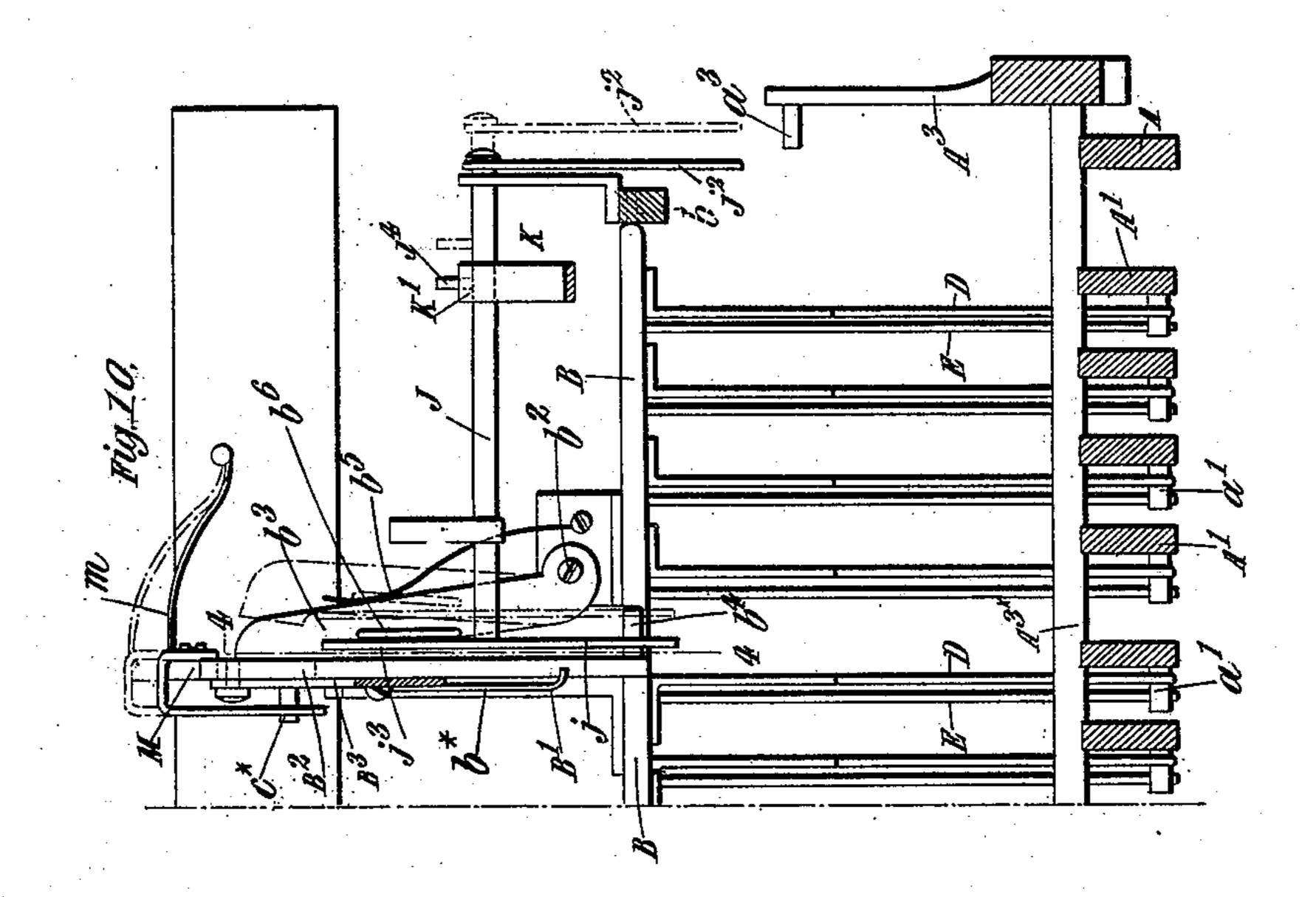
(Application filed Oct. 2, 1899.)

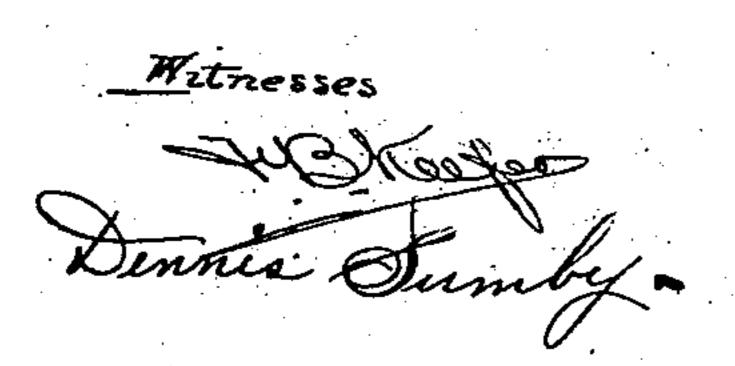
(No Model.)

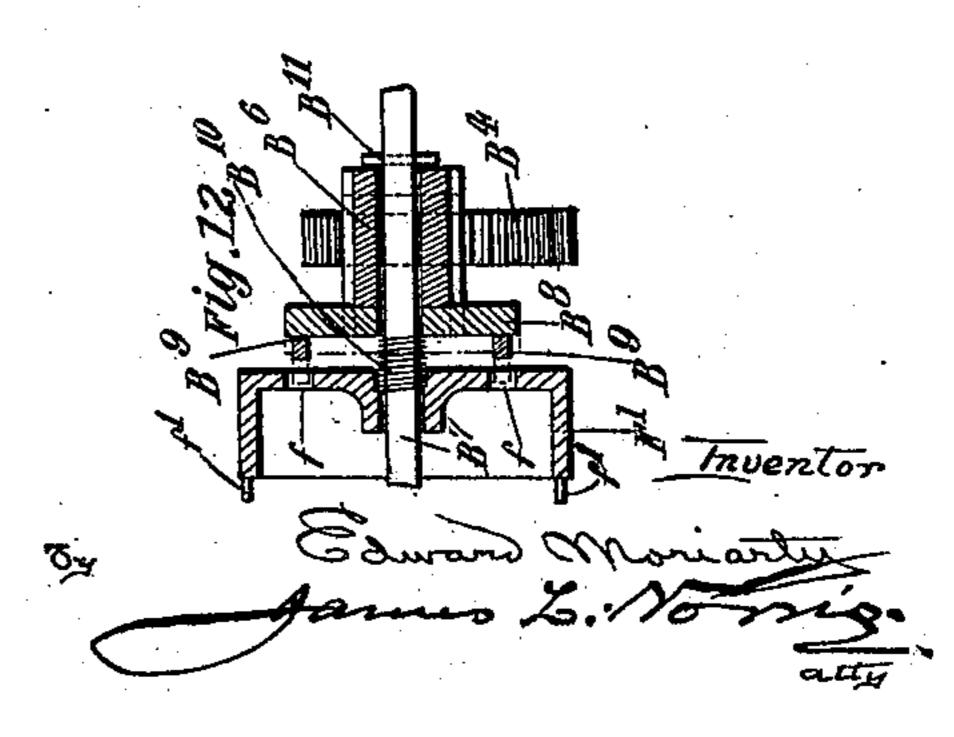










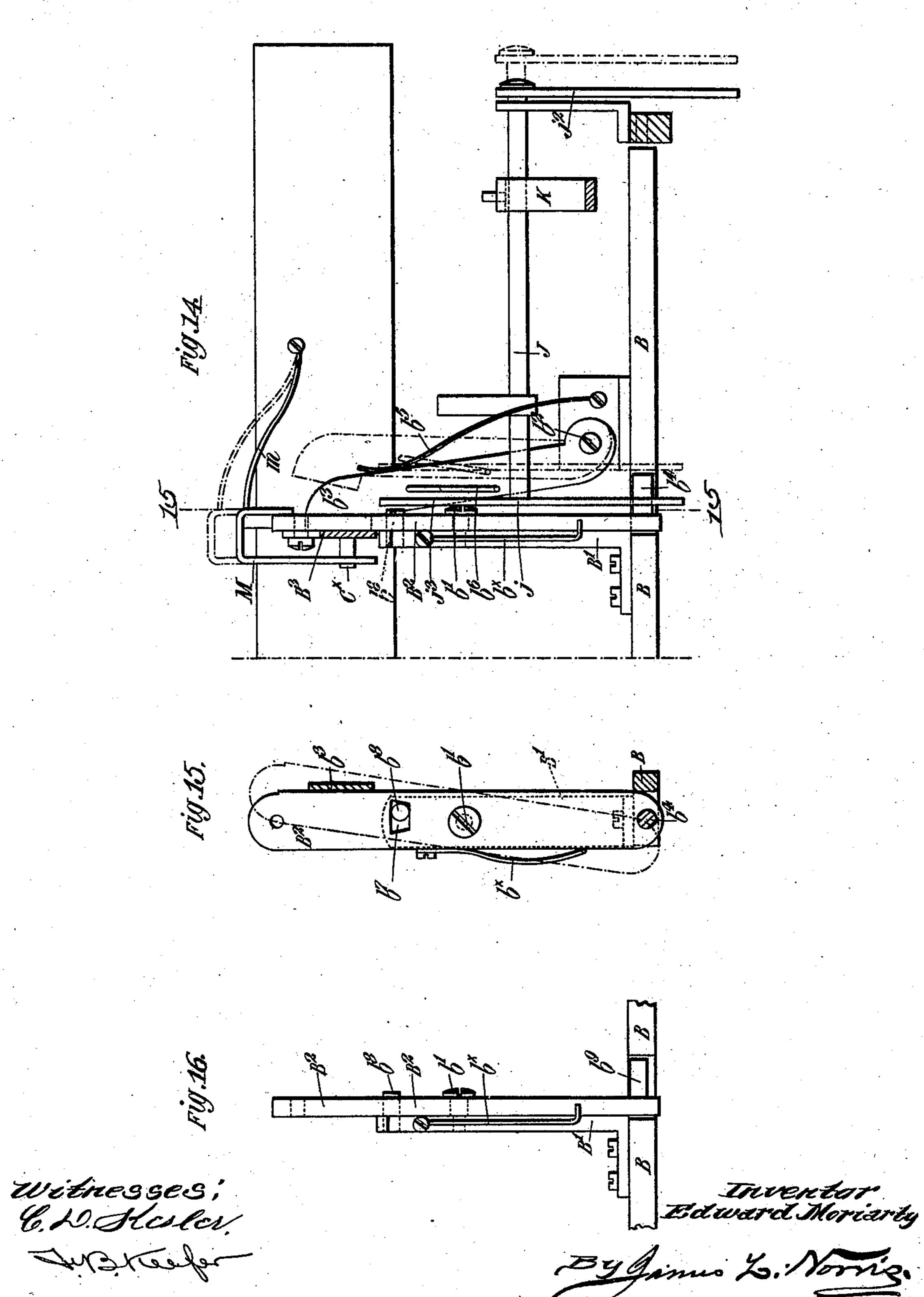


#### COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

(No Model.)

7 Sheets—Sheet 6.

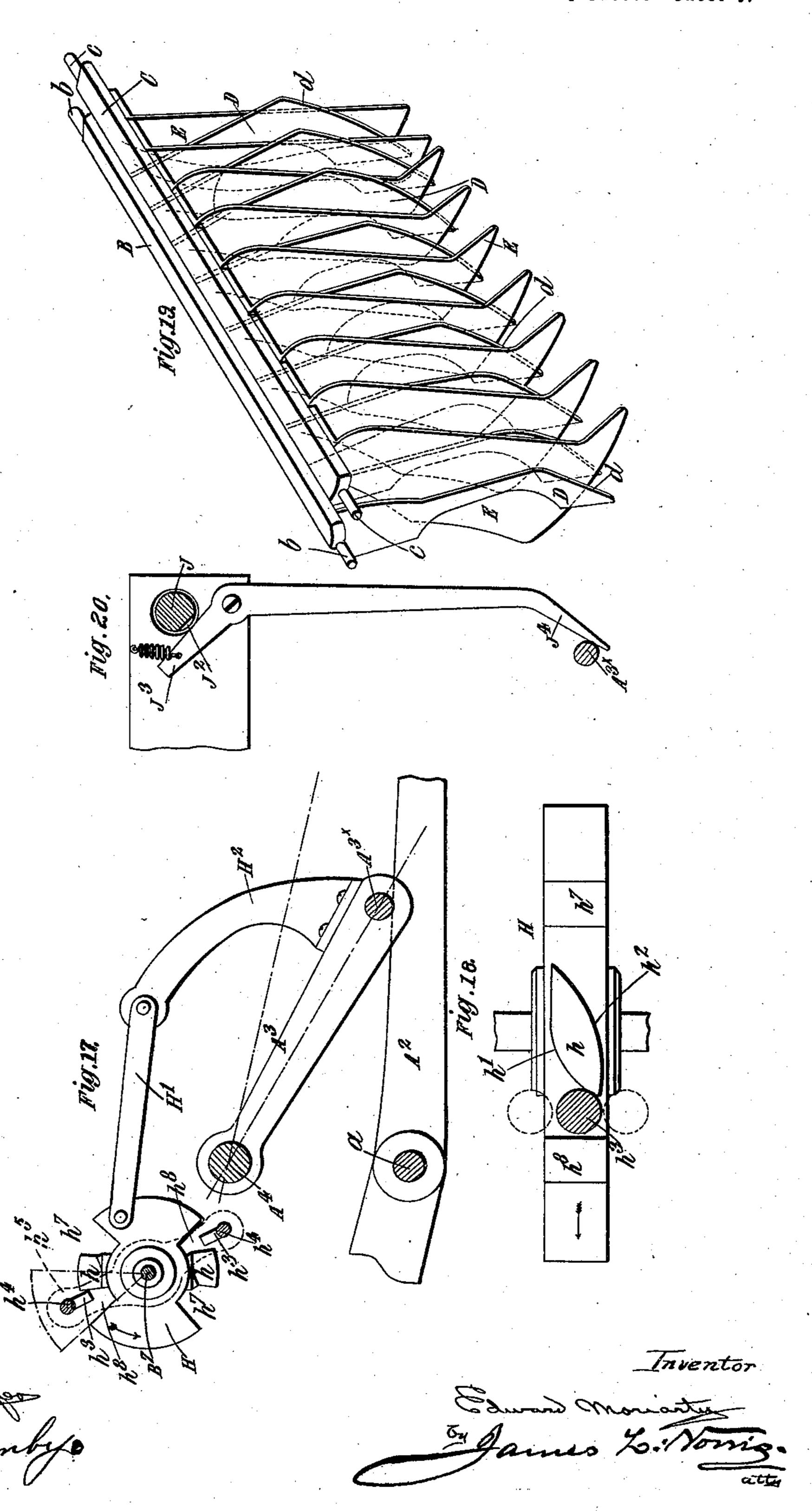


# E. MORIARTY. COUNTING OR ADDING APPARATUS.

(Application filed Oct. 2, 1899.)

(No Model.)

7 Sheets—Sheet 7.



# United States Patent Office.

#### EDWARD MORIARTY, OF FULHAM, ENGLAND.

#### COUNTING OR ADDING APPARATUS.

SPECIFICATION forming part of Letters Patent No. 689,738, dated December 24, 1901.

Application filed October 2, 1899. Serial No. 732, 392. (No model.)

To all whom it may concern:

Be it known that I, EDWARD MORIARTY, beer retailer, a subject of the Queen of Great Britain, residing at 180 North End road, Ful5 ham, in the county of Middlesex, England, have invented certain new and useful Improvements in Counting or Adding Apparatus, of which the following is a specification.

This invention relates to counting or addio ing apparatus which is specially intended for
use with money-tills of the kind in which
there are employed a series of key-levers bearing numerals or other signs to correspond
with the various amounts of money the till is
intended to deal with, such key-levers working in conjunction with counting or registering mechanism, whereby the various amounts
of money dealt with are added together and
recorded by registering-drums, so that the
total amount of money which should be found
in the till is represented by the said drums.

According to my invention I provide in proximity to the inner ends of the aforesaid key-levers and in a position to be operated 25 thereby a number of plates formed with inclined or cam-shaped edges or portions and hereinafter referred to as "cam-plates," a separate cam-plate being provided for each of the aforesaid key-levers. These cam-plates are 30 arranged in series, each series being preferably carried by a rocking bar or shaft. The cam-plates of each series are of different contour—that is to say, their shape is such that when the key-levers are actuated the extent 35 of rocking movement imparted to the said rocking bar will be greater or less, according as a key-lever representing a higher or a lower denomination is actuated.

Means are provided for transmitting the movement of the cam-plates to the registering-drums, which are caused to move angularly to an extent equivalent to the amount of movement which the said rocking bars receive from their cam-plates when the latter are actuated by the key-levers. Means are also provided for transferring from one registering-drum to the next of higher denomination the amount beyond that which can be registered on the drum of the lower denomination, such means comprising a movable arm carried by each of the rocking bars, which arm

is released and permitted to move independ-

ently of the rocking bar when a transfer is to be made, means being also provided whereby the extent of movement of said movable 55 arm is controlled. Means are also provided whereby the extent of movement of the aforesaid rocking bars cannot exceed that which is necessary for effecting any particular recording stroke, no matter how sharply or 60 suddenly the key-levers may be struck in actuating them. For this purpose I employ stop-pieces carried by additional rocking bars, having cam-plates which are in form duplicates of the cam-plates on the main 65 rocking bars with the exception that they are arranged in reverse order. Both of these sets of cam-plates and their rocking bars are adapted to move simultaneously, but in opposite directions.

In order that my invention may be clearly understood, I will proceed to describe the same more fully with reference to the accompanying drawings, which illustrate a moneytill of the kind hereinbefore referred to provided with my improved counting or adding apparatus.

Figure 1 is a front elevation of the complete money-till. Fig. 2 is a similar view with the front part of the casing removed. Fig. 80 3 is an end view of the money-till with the end part of the casing removed. Fig. 4 is a front elevation showing the various registering-drums and parts immediately in connection therewith comprising the registering de-85 vice. Fig. 5 is a transverse section in the line 5 5, Fig. 4, looking toward the right. Figs. 6 and 7 are respectively an end view looking toward the right and a front elevation showing, on a larger scale, the five-cents 90 drum and the adjacent end of the transferring-lever which the drum operates. Fig. 8 is a plan view of the internal mechanism. Fig. 9 is a transverse section in the line 9 9, Fig. 8, drawn on a larger scale. Fig. 10 95 is a longitudinal section in the line 10 10, Fig. 8, also drawn on a larger scale, showing the transferring mechanism hereinafter described. Fig. 11 is a detail sectional view taken approximately in the line 1111, Fig. 8, 100 and drawn on an enlarged scale. Fig. 12 is a vertical section in the line 12 12, Fig. 9. Fig. 13 is a transverse section taken approximately in the line 13 13 of Fig. 11. Fig. 14

is a view similar to Fig. 10, but showing simply the upper part of said last-mentioned figure and on a larger scale. Fig. 15 is a vertical section taken approximately in the line 5 15 15 of Fig. 14 and looking toward the left. Fig. 16 is a front elevation of certain of the parts illustrated in Fig. 14 and hereinafter more fully described. Fig. 17 is a detail sectional view taken approximately in the line 10 17 17 of Fig. 8 and drawn on an enlarged scale. Fig. 18 is a plan view of a cam hereinafter more fully referred to. Fig. 19 is a perspective view of the main and additional rocking bars and their cam-plates. Fig. 20 15 is a transverse section taken in the line 2020 of Fig. 8.

Like letters of reference indicate similar

parts in all the figures.

A A' A² are the key-levers, which are of the ordinary construction usual in this class of apparatus—that is to say, they consist of bars pivoted at α to the framing of the apparatus and maintained with their outer ends elevated by a rocking frame A³, hinged to a transverse rod A⁴ and normally kept in its depressed position by a spring A⁵₀, (see Fig. 8,) connected, respectively, to the rocking frame and the framework. Other springs, as A⁵¹, Fig. 3, may bear against pins, as A⁵²,

on the key-levers. The said outer ends of the key-levers bear figures or other signs designating the amounts of money which the various key-levers represent. In the example illustrated the key-levers are arranged in

three series or divisions—viz., the key-lever A, representing five cents, the key-levers A', representing, respectively, ten, twenty, thirty, forty, fifty, sixty, seventy, eighty, and ninety cents, and the key-levers A<sup>2</sup>,

40 representing, respectively, one, two, three, four, five, six, seven, eight, and nine dollars. It will therefore be seen that this arrangement comprises three divisions of the key-levers—one key-lever A, a series of nine

45 key-levers A', and another series of nine key-levers A<sup>2</sup>. Obviously, however, this arrangement may be varied in any desired manner to accord with the requirements of the apparatus. The inner ends of the key-levers 50 work in vertical slots A<sup>6</sup>, Fig. 9, at the back

of the machine, which slots serve the double purpose of guiding said key-levers and of limiting the extent of their vertical movement.

B B are the main rocking bars of which

B B are the main rocking bars, of which there are two in the present example, one for the series of key-levers A' and the other for the series of key-levers A<sup>2</sup>.

C C are the additional rocking bars, arranged adjacent to and parallel with the 60 aforesaid bars B. These bars B and C are each provided with pivots b and c, respectively, at their ends, which pivots are mounted in suitable bearings forming part of the framing of the apparatus.

D D are the cam-plates carried by the main rocking bars B, and E E are the cam-plates carried by the additional rocking bars C.

F F' F<sup>2</sup> F<sup>3</sup> F<sup>4</sup> are the registering-drums, the drum F bearing alternately the figures "0" and "5" and the drum F' bearing the figures 70 "0, 10, 20, 30, 40, 50, 60, 70, 80, 90." The drums F<sup>2</sup>, F<sup>3</sup>, and F<sup>4</sup> bear figures to represent units, tens, and hundreds, respectively.

The series of cam-plates D depend from the rocking bar B and are connected thereto, so 75 that they all move simultaneously therewith. Each cam-plate is formed with a differentlyshaped cam-surface d, so that the extent of rocking movement imparted to the rocking bar B when one or other of the cam-plates is 80 actuated by a key-lever will vary to a predetermined extent, according as one or other of the cam-plates is actuated—that is to say, if a key-lever representing the lowest denomination of the series of key-levers A' be oper- 85 ated the said rocking bar B will be rocked to a definite predetermined extent; but if another key-lever of the same series be operated representing a denomination, say, twice or thrice as much as the key-lever of lowest de- 90 nomination then the extent of rocking movement of the said bar will be correspondingly twice or thrice as great. The two series of cam-plates carried by the two bars BC are so arranged that they lie side by side (see Fig. 95 19) with their cam-surfaces facing in opposite directions, so as to form pairs of camplates. Each of the key-levers is provided with a lateral pin a', Fig. 9, which lies immediately below the various pairs of cam- 100 plates, and when the outer ends of the keylevers are depressed these pins rise and enter between the pairs of cam-plates, thereby causing them to be shifted in opposite direcrections and the rocking bars to likewise be 105 rocked in opposite directions, as shown by the dotted lines in Fig. 9.

Each of the rocking bars B is provided with an arm or bracket B', to which is pivotally connected at b' a movable arm  $B^2$ , which is 110 normally prevented from turning about its pivot by a locking device, hereinafter described. The said arm B2 is connected at its outer end by a connecting-rod B3 to a segmental rack B4, mounted on a transverse rod 115 B<sup>5</sup>. This rack gears with an elongated pinion B<sup>6</sup>, loosely mounted on a transverse shaft B<sup>7</sup>. The said elongated pinion forms part of a clutch device B8, which in the example illustrated comprises a pair of arms having at 120 their extremities lateral pins B9, lying adjacent to the face of a registering-drum. The registering-drum is formed with a concentric series of holes f, corresponding in number to that of the divisions or denominations which 125 the drum bears, and with these holes the pins B<sup>9</sup> are adapted to engage by a lateral movement of the clutch device.

B<sup>10</sup> is a spring interposed between each of the registering-drums and the said clutch de- 130 vices, such spring normally tending to keep the clutch device pressed away from its adjacent drum against a stop-piece B<sup>11</sup> on the shaft B<sup>7</sup>. The pins B<sup>9</sup> of the clutch device

are thus kept out of contact with the drum, i although they lie sufficiently close thereto to insure their quickly engaging with the holes f in the drum when the clutch device is lat-5 erally shifted by the stop-piece B<sup>11</sup> on the shaft as the latter is moved longitudinally by the action of a cam H. This cam is concentrically arranged with respect to the aforesaid shaft B<sup>7</sup> and is connected, by means of 10 a rod H', Fig. 17, to an arm H<sup>2</sup>, forming part of the rocking frame A<sup>3</sup>, so that each time a key-lever is operated and the said frame caused to swing about its bar A4 the cam will be thereby oscillated. The said cam is formed 15 with radial portions hh, arranged diametrically opposite to one another, and each portion is formed with two cam-surfaces h'  $h^2$ , Fig. 18. These cam-surfaces cooperate with pins  $h^3$   $h^3$ , carried by rods  $h^4$ , extending from arms 20  $h^5$ , that are fixed radially to the said shaft  $B^7$ . The aforesaid rods  $h^4$  are adapted to slide in guide-pieces  $h^6$ , which serve to steady the pins  $h^3$   $h^3$  while they are being acted upon by the said cam. When one or other of the key-25 levers is depressed, its initial movement lifts the aforesaid swinging frame A3, which in turn imparts angular movement to the cam in the direction represented by the arrow in Fig. 17. This movement of the cam causes its cam-sur-30 faces h', Fig. 18, to act upon the pins  $h^3$ , which consequently slide the shaft B7 longitudinally from right to left of the machine and bring the clutch devices into engagement with their adjacent drums. By the continued down-35 ward movement of the key-lever the said rocking bars are actuated and cause the segmental rack B4 to turn angularly to an extent depending on the denomination of the keylever operated. This angular movement of 40 the segmental rack turns the elongated pinion and clutch device to a corresponding extent, whereby the registering-drum belonging to the series of cam-plates under operation is caused to turn a sufficient distance to effect 45 its registering movement. By the time the key-lever has been fully depressed and the cam H has completed its movement in the direction of the said arrow the aforesaid pins  $h^3$  arrive in the spaces  $h^7$  of the cam H, where-50 by they escape from the cam-surfaces h' and permit the springs B<sup>10</sup> of the clutch devices to shift the said shaft B7 from left to right of the machine and return it to its original position, thus carrying the pins B9 out of en-55 gagement with the holes in the registeringdrums. Then when the key-lever is liberated and allowed to return to its original position under the action of the spring-controlled rocking frame A<sup>8</sup> the cam H is by the latter 60 shifted in the reverse direction, whereby the other cam-surfaces  $h^2$  of said cam operate on the aforesaid pins  $h^3$ . By so doing they cause the shaft B7 to move longitudinally still farther in the direction of left to right of the 65 machine against the resistance of a spring  $B^{12}$ , whereby the pins  $h^3$  are positively carried

I thus avoid any liability of the pins engaging with the drums during the time that the elongated pinions are being revolved in the 7c reverse direction by their segmental racks as the latter return to their normal position under the influence of springs  $B^{\times}\,C^{\times}$  on the rocking bars B C and springs B<sup>5×</sup> on the transverse rods B<sup>5</sup>. As soon as the said pins es- 75 cape from the cam-surfaces  $h^2$  and arrive opposite the space  $h^8$  of the cam the reaction of the aforesaid spring  $B^{12}$  shifts the shaft  $B^7$ longitudinally in the reverse direction—i. e., from right to left—so as to bring the pins  $h^3$  80 into their normal position, ready to be again acted upon by the cam-surfaces h' when a key-lever is next depressed. At the same time the pins of the clutch devices are brought into the aforesaid position of close proximity 85 to the registering-drums, ready to immediately reëngage with the holes in said drums when a next recording operation is to be performed. In order to prevent the aforesaid spring B<sup>12</sup> from carrying the shaft B7 too far toward the 90 left, the end of said spring is adapted to strike

against a stop B<sup>13</sup>.

The aforesaid registering-drums are provided with appropriate means for retaining them in any of the angular positions to which 95 they are set by the apparatus. In the drawings I have shown the drums F' and F<sup>2</sup> provided with lateral pins f', whose number corresponds with the number of steps which the said drums perform in completing a single 100 revolution. Engaging with these pins are two pawls  $f^2 f^3$ , arranged contiguous to each of the drums, the pawl  $f^2$  serving to prevent the drum from moving in a forward direction and the pawl  $f^3$  to prevent it from moving in 105 a rearward direction. The pawls  $f^2$  are carried by a spindle  $f^4$ , which is furnished with a spring  $f^5$  for causing the pawls to engage with the pins. Near one end of said spindle is a finger  $f^6$ , which lies contiguous to a fin- 110 ger  $f^7$ , carried by another spindle  $f^8$ . This last-mentioned spindle has a lug or projection  $f^9$ , which lies in the path of an arm  $A^5$ , carried by the rocking frame A<sup>3</sup>, so that each time the said rocking frame is elevated when 115 a key-lever is depressed the arm A<sup>5</sup> acts upon the aforesaid lug or projection  $f^9$  and so turns the spindles  $f^8$  and  $f^4$  in a direction to remove the pawls from engagement with the pins on the drums. This movement takes place im- 120 mediately a key-lever commences to descend and before the aforesaid clutch devices B<sup>8</sup> have had time to engage with the drums. The pawl  $f^3$  is maintained in engagement with the pins f' by a spring  $f^{10}$ , which per- 125 mits said pawl  $f^3$  to give way to the pins f' as the drum moves in a forward direction—i.e., in the direction of the arrow in Fig. 5—but does not allow said pawl to give way to the pins if the drum tends to move in the oppo- 130 site direction, the shoulder  $f^{11}$  then acting as a stop.

As the drum F (which registers alternately completely away from the registering-drums. I five cents and zero) does not require to be

turned more than one step at a time, I do not employ a cam-plate and rocking bar for operating it, but employ means for actuating it directly from its key-lever A. For this 5 purpose I provide said drum with a springpawl I, Figs. 11 and 13, to engage with a ratchet i, forming part of a sleeve i'. This sleeve is provided with a radial arm  $i^2$ , which is connected by a link  $i^3$  to the key-lever A. 10 The drum and the aforesaid sleeve are both loosely mounted on their shaft, and therefore each time the key-lever A is depressed the said drum is caused to travel a distance of one step through the intervention of the pawl 15 and ratchet, as is well understood. In order to prevent said drum from being unintentionally turned in either direction, I provide a spring-pawl I', which is adapted to engage with lateral pins I<sup>2</sup> on the drum. The for-20 mation of this pawl is such that the tendency of the drum to turn forwardly when the keylever A is depressed causes the pins to raise said pawl; but any tendency for the drum to turn in the reverse direction is overcome by 25 a shoulder I3 on said pawl. The aforesaid movable arm B2 is, as already stated, capable of receiving movement about its pivot b' independently of the movement it receives from the main rocking bar B. Nor-30 mally, however, the said arm B2 is prevented from turning about its pivot by a locking device. When it is necessary for the machine to transfer the amount recorded by one registering-drum to another of higher denomi-35 nation, the said locking device is thrown out of action and the independent movement of the arm B2 is effected by a contrivance which is actuated by the registering-drum of lower denomination. The contrivance illustrated 40 in the drawings for releasing the said arm B2 and for effecting the independent movement thereof comprises a shaft J, Figs. 9 and 10, which is capable of sliding longitudinally and of oscillating. It is mounted in bearings that 45 are independent of the aforesaid main rock-

ing bar, and at its inner end it is provided

with a crank or arm j, having an elongated

slot j' therein to engage with a pin  $b^4$  on the

inner end of the movable arm B2. The outer

ger or cam piece j2, which so long as the said

shaft occupies its inward position (represent-

ed by the full lines in Fig. 10) lies out of the

path of a pin or projection  $a^3$  on the rocking

prises a plate or finger  $b^3$ , which is pivotally

connected at  $b^2$  to a bracket on the rocking

bar B and by a spring  $b^5$  is normally main-

tained in a position transverse of the plane

the latter cannot at such time be shifted in

the direction of said locking device or, in other

words, cannot perform its aforesaid independ-

ent movement. The said locking device is

adapted to be acted upon by an arm  $j^3$  on the

65 provided with an elongated pin  $b^6$ , which is

60 of movement of the movable arm B2, so that

55 frame A<sup>3</sup>. The aforesaid locking device com-

50 end of the said shaft J is provided with a fin-

higher denomination to turn one step and add to its record ten cents transferred from the drum F. K<sup>3</sup> is a spring which acts to return the aforesaid lever K and the shaft J to their normal 115 position after each displacement by the pins  $k^2$ , and  $b^{\times}$  is a spring for returning the arm B<sup>2</sup> to its normal position. The end K<sup>2</sup> of lever K is provided with a cam-piece or incline K4, (represented best in Fig. 15,) which lies 120 in the path of the pins  $k^2$ , so that as the drum F moves in the direction of the arrow shown in Fig. 16 they one by one come against the inclined portion K4 of the lever and shift the end adjacent to the drum outwardly, the said 125 lever K at this time turning on the pivot k. In the case of transferring the records of

the drum F' to the drum F2 the lever K for

sliding the shaft J on the main rocking bar

sake of convenience in construction situated

at some distance from the drum F'. There-

fore it has been necessary to provide a de-

aforesaid shaft J when the latter is caused to I vice between said drum and said lever K for

of the dollars series of key-levers A2 is for 13c

slide longitudinally into its outward position.

By the outward sliding movement of the said

said elongated pin  $b^6$  and the arm  $j^3$  with-

drawn from its restraining position, so that

the arm B<sup>2</sup> is free to turn about its pivot and

to perform its independent movement. For

ment to the said shaft J at the proper times,

a lever K, pivoted at k, is provided, the end

K' of such lever engaging with a pin  $j^4$  on the

said shaft J and the other end K2 lying in a

registering-drum or by a device (hereinafter

described) which is under the control of said

pins. The aforesaid movable arm B2 is pro-

vided with a curved slot  $b^7$ , into which pro-

pin serves as a stop for limiting the extent of

the independent movement which the said

arm B<sup>2</sup> can perform. In the case of the five-

cents drum F, which alternately registers five

fer from this drum to the next drum of higher

denomination—viz., to the drum F'—would

be necessary at each alternate step of the

drum F. In the drawings the five-cents drum

plete revolution, and therefore it is provided

with three lateral pins  $k^2$  at a distance apart

of two steps. At each alternate operation of

the five-cents key-lever A one of the pins  $k^2$ 

lever K, which thus shifts the sliding shaft

J outwardly into a position to bring its cam-

piece  $j^2$  in the path of the aforesaid pin  $a^3$  on

the rocking frame A<sup>3</sup>. Then at the next suc-

A the said shaft J will be rocked by the said

pin  $a^3$  striking the cam-piece  $j^2$ , and will there-

by shift the movable arm B2 (through the in-

tervention of the crank j and the pin  $b^4$ ) a

acts upon the inner end K2 of the aforesaid 100

ceeding operation of the five-cents key-lever 105

sufficient distance to cause the drum F' of 1:0

F illustrated performs six steps at each com- 95

cents and zero, as already explained, a trans- 90

jects a pin  $b^8$  on the arm or bracket B'. This 85

position to be acted upon by pins  $k^2$  on the 80

the purpose of imparting the sliding move- 75

shaft J the locking device is by means of the 70

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enabling the latter to be operated by the pin  $k^2$  on the drum F'. For this purpose I have indicated a spindle L, having at its end adjacent to the drum F' a finger l, which lies in the path of the said pin  $k^2$  of this drum. At the opposite end of said spindle L is a pin l', engaging with the slotted end  $k^3$  of the aforesaid lever K, so that each time said bar L is actuated it imparts movement to the lever K, which in turn imparts sliding movement to the shaft J, and thereby brings its campiece  $j^2$  into position to be acted upon by the projection  $a^3$  of the rocking frame  $A^3$ .

The aforesaid additional rocking bars C are each provided with a stop-piece C', which lies directly behind the arm B<sup>2</sup> of the rocking bar B, so that when any of the key-levers of the series A' A<sup>2</sup> are depressed the said arms on the additional rocking bars will be caused to approach the arms B<sup>2</sup> on the main rocking

bars.

As already stated, the rocking bars C are provided with cam-plates which are arranged in the reverse order of that in which those 25 on the bars B are arranged, so that although both the bars B and C rock simultaneously in opposite directions the extent of movement of the bars C is greatest when that of the bars B is least, and vice versa. By these 30 means I am able to vary the position of the stop-piece C relatively to the arms B2, so that said stop-pieces will assume a position each time a key-lever of the series A' or A2 is actuated to insure that the arms B2 cannot shift 35 beyond the distance necessary to cause the drums F' F2 to correctly register the amount represented by the key-lever depressed. This prevents any liability of the registeringdrums from being turned beyond their proper 40 distance no matter how suddenly or forcibly

a key-lever is depressed. To prevent the arms B2 from shifting beyond their proper ambit of movement when they are operated independently of the main 45 rocking bar, as aforesaid, I arrange immediately at the rear of the upper ends of said arms B2 a hinged stop-piece M, which is maintained in this position by a spring m, the distance between said hinged stop-piece and the 50 said arm B2 being only sufficient to enable the latter to perform its proper transfer-stroke. The said hinged stop-piece is provided with an inclined or cam surface m' for a pin  $c^{\times}$  on the stop-piece C' of the additional rocking bar 55 C to act upon as said stop-piece C' advances toward the arms B<sup>2</sup>. The hinged stop-piece M is thus raised into a position out of reach of the arm B<sup>2</sup> when the latter is moved with

the main rocking bar B, and it therefore does to not impede this movement.

In some cases a transfer from one drum to another has to be effected by the machine simultaneously with its ordinary recording operation—as, for instance, when a record from the five-cents drum F has to be transferred to the drum F' at the same time as, say, twenty cents has to be recorded by this drum. This

would require the latter drum not only to be turned two steps for registering the twenty cents, but also another step to register the 7° ten cents transferred from the drum F. In this case the bars B and C would be rocked by their cam-plates, as already stated, and impart to the drum F' the necessary movement for registering twenty cents. At the 75 same time the arm B2 would be rocked independently by the shaft J as the latter was actuated by the pin  $a^3$  operating on the campiece j². An augmented movement would therefore be given to the said drum F' for 80 adding to its record the extra ten cents transferred from the drum F. The removal of the plate b3 from between the arms B2 and C' permits of the extra amount of movement of the arm B2, the thickness of said plate being just 85 sufficient to leave a space between said arms B<sup>2</sup> and C' wide enough for this movement, and no more. The said arm C' thus still serves as a stop-piece (even when the transferring is being effected) to prevent move- 90 ment beyond that which is necessary for ef-

fecting the proper record.

In view of the fact that the shaft J of the series of key-levers A2 is in the example illustrated situated in an elevated position and 95 that therefore the pin  $a^3$  on the rocking frame A<sup>3</sup> cannot actuate the finger or cam piece j<sup>2</sup> thereon until the said frame A3 reaches its uppermost position it is necessary to provide means whereby the return of this shaft J to 100 its original position by the action of the spring will not take place immediately the lever L has been actuated by the pin  $k^2$ , but will be retarded until after the said shaft has been rocked by the said pin  $a^3$  and cam-piece  $j^2$  to 105 effect the transferring operation. For this purpose I provide the said shaft with a notch J<sup>2</sup>, Fig. 20, with which a spring-controlled arm J<sup>3</sup> engages each time the said sliding rod is shifted into its outward position. This 110 spring-controlled arm is provided with a tailpiece J4, against which the transverse bar A3X of the aforesaid frame A<sup>3</sup> strikes when it assumes its lowermost position, and the eby causes the arm J<sup>3</sup> to become disengaged from 115 the notch in the shaft J. The said shaft being thus released returns to its normal position after having been actuated by the said pin  $a^3$  and cam-piece  $j^2$ . It will be obvious, however, that no such device for retarding 120 the return movement of the sliding shaft will be necessary when the construction of the machine permits the cam-piece j² to be arranged in a position relatively to the pin  $a^3$ similar to that occupied by the cam-piece j2 125 and pin  $a^3$  of the device for transferring the records from the drum F to the drum F'.

The drums F<sup>3</sup> and F<sup>4</sup> may be operated from the drum F<sup>2</sup> in any convenient and ordinary manner, such means forming no part of my 130 present invention.

What I claim is—

1. In adding apparatus, the combination with the series of key-levers and the series of

cam-plates adapted to be actuated thereby, of a registering-drum, of a longitudinally-movable shaft upon which said registering-drum is loosely mounted, a device capable of engaging with said drum when the shaft is shifted longitudinally in the proper direction, of means for longitudinally shifting said shaft and of means for transmitting to said engaging device the varying movements of the aforesaid cam-plates for the purpose specified.

2. In adding apparatus, the combination with the series of key-levers and the series of cam-plates adapted to be actuated thereby, of a registering-drum, a longitudinally-movable shaft upon which said registering-drum is loosely mounted, of a clutch device for engaging with said drum and loosely mounted on said shaft but adapted to move longitudinally therewith, of means for longitudinally shifting said shaft, of an elongated toothed pinion forming part of and moving with said clutch device, of a rack in constant gear with said pinion, and of means for transmitting to said rack the varying movements of the aforesaid cam-plates, for the purpose specified.

3. In adding apparatus, the combination with the series of key-levers and the series of cam-plates adapted to be actuated thereby, of a registering-drum, a longitudinally-mov-30 able shaft upon which said registering-drum is loosely mounted and formed with a circular series of apertures in one of its faces, of a clutch device consisting of arms having pins at their extremities for engagement with said 35 apertures in the registering drum, the said clutch device being loosely mounted on said shaft but capable of longitudinal movement therewith, of means for longitudinally shiftingsaid shaft, of an elongated toothed pinion 40 forming part of and moving with said arms, of a segmental rack in constant gear with said pinion, a bar which is common to the series of cam-plates, and of a connecting-rod coupled respectively to the said segmental 45 rack and to said bar for the purpose specified.

4. In adding apparatus, the combination with the registering-drum, a longitudinally-movable shaft loosely carrying said drum, and with the means for transmitting to said drum the varying movements of the cam-plates when the key-levers are actuated; of retaining-pawls for normally retaining the said drum in a fixed position, and of means for releasing one of said pawls to permit of the drum's rotary movement by the aforesaid motion-transmitting means substantially as described.

5. In adding apparatus, the combination with the registering-drum, a longitudinally60 movable shaft loosely carrying said drum, and with cam-plates, key-levers, and the means for transmitting to said drum the varying movements of the cam-plates when the key-levers are actuated; lateral pins on said drum, of two retaining-pawls adapted to engage said lateral pins, one of the pawls acting to prevent movement of the drum in a forward di-

rection and the other acting to prevent movement of the drum in a backward direction, whereby such pawls normally retain the said 70 drum in a fixed position, of a spindle to which the pawl preventing forward movement of the drum is connected, and of means for actuating said spindle to release the pawl it carries each time a key-lever is operated substantially 75 as described.

6. In adding apparatus the combination with the longitudinally-movable shaft and the registering-drum having lateral pins mounted loosely on the longitudinally-movable shaft 80 and with cam-plates, key-levers and the means for transmitting to said drum the varying movements of the cam-plates when the keylevers are actuated, of two retaining-pawls adapted to engage with the lateral pins on said 85 drum, one of said pawls acting to prevent movement of the drum in a forward direction and the other acting to prevent movement of the drum in the reverse direction, whereby such drum is normally retained in a fixed po- 90 sition, of a spring-controlled spindle to which the pawl preventing forward movement of the drum is connected, of a finger on said spindle, of a rocking frame which latter receives motion each time one or other of the key-le- 95 vers is actuated and of means for actuating said spindle each time the said rocking frame is operated substantially as and for the purpose specified.

7. In adding apparatus, the combination roo with the longitudinally-movable shaft and the registering-drum having lateral pins mounted loosely on the longitudinally-movable shaft and with cam-plates, key-levers and the means for transmitting to said drum the varying 105 movements of the cam-plates when the keylevers are actuated, of two retaining-pawls adapted to engage with the lateral pins on said drum, one of said pawls acting to prevent movement of the drum in a forward direction 110 and the other acting to prevent movement of the drum in the reverse direction, whereby such drum is normally retained in a fixed position, of a spring-controlled spindle to which the pawl preventing forward movement of the 115 drum is connected, of a finger on said spindle, of a rocking frame which latter receives motion each time one or other of the key-levers is actuated, of a second spindle provided with a finger adjacent to the finger on the first spin- 120 dle, of an arm on said second spindle and of a projection on the rocking frame for actuating said arm, substantially as and for the purpose specified.

8. In adding apparatus, the combination with the longitudinally-movable shaft and the registering-drum and the clutch device both mounted loosely on the longitudinally-movable shaft, and with a key-lever and the means for imparting rotary movement to said clutch device; of a cam adapted to be actuated each time a key-lever is operated, and of projections on the aforesaid longitudinally-movable shaft upon which projections said cam oper-

ates when actuated and thereby imparts the said longitudinal movement to the shaft and the said clutch device thereon, for the pur-

pose specified.

9. In adding apparatus, the combination with the longitudinally-movable spring-actuated shaft and the registering-drum and the clutch device both mounted loosely on the longitudinally-movable shaft, and with a key-10 lever and the means for imparting rotary movement to said clutch device; of a cam located concentrically with respect to the longitudinally-movable shaft, of means for imparting to said cam an oscillatory movement 15 each time a key-lever is operated, and of projections on said shaft with which the cam engages to impart a longitudinal movement to said shaft first in one direction and then in the opposite direction, thereby alternately 20 causing the said clutch to engage and disengage the said registering-drum for the purpose specified.

10. In adding apparatus, the combination with the longitudinally-movable shaft and the 25 registering-drum and the clutch device both loosely mounted on the longitudinally-movable shaft and key-lever and with the means for imparting rotary movement to said clutch device; of a cam located concentrically with 30 respect to the longitudinally-movable shaft and having a double cam-surface, one portion of which operates to impart to the said shaft longitudinal movement in one direction and the other portion of which operates to 35 impart to the said shaft longitudinal movement in the opposite direction, of means for imparting to said cam an oscillatory movement each time a key-lever is operated, of projections on said shaft for the said double 40 cam-surface to act upon, and of springs for operating to return said shaft to its normal po-

sition for the purpose specified.

11. In adding apparatus, the combination with the longitudinally-movable shaft and the 45 registering-drum and the clutch device both loosely mounted on the longitudinally-movable shaft and having a pair of double camsurfaces disposed diametrically opposite each other, a rocking frame, of a connecting-rod 50 coupling the said cam to a rocking frame, said rocking frame being lifted each time a key-lever is depressed, thereby imparting an oscillatory movement to said cam, arms on said longitudinally-movable shaft, of a pair of lat-55 eral projections extending from said arms and bearing pins at their extremities, which pins are acted upon by said cam-surfaces to impart the longitudinal movement to the said movable shaft, and of springs acting in oppo-60 site directions upon said shaft substantially as and for the purpose specified.

12. In adding apparatus, the combination with the series of key-levers, the series of camplates and the registering device, of an addi-65 tional series of cam-plates adapted to be operated simultaneously with the main series of cam-plates to prevent the latter from ex-

ceeding their proper ambit of movement when a key-lever is actuated.

13. In adding apparatus, the combination 70 with the series of key-levers, the main series of cam-plates, the rocking bar carrying them, and the registering device, of an additional series of cam-plates carried by a rocking bar and adapted to be operated by the said key- 75 levers simultaneously with the operation of the main series of cam-plates to an extent depending on that to which the main cam-plates are operated, and of stop-pieces on the rocking bars of the said additional and main cam- 80 plates respectively, for the purpose specified.

14. In adding apparatus, the combination with the series of key-levers, the main camplates, the rocking bar carrying them and the registering device, of an additional series of 85 cam-plates carried by a rocking bar and adapted to be operated by the said key-levers simultaneously with the operation of the main series of cam-plates to an extent depending on that to which the main cam-plates are op- 9° erated, and of stop-pieces on the rocking bars, one of such stop-pieces being in the form of a movable arm carried by the main rocking bar and the other in the form of a fixed arm carried by the additional rocking bar sub- 95 stantially as and for the purpose specified.

15. In adding apparatus, the combination with the series of key-levers, the series of main cam-plates, the rocking bar carrying them, and the registering device, of an additional 100 series of cam-plates carried by a rocking bar, the cam-surfaces of the latter series of camplates being situated in close proximity to the cam-surfaces of the main series of camplates and facing in an opposite direction 105 thereto, and of projections on the aforesaid key-levers adapted to enter between the proximate cam-surfaces of the two series of cam-plates each time the key-levers are actuated, whereby the said two series of cam- 110 plates are shifted in opposite directions for

the purpose specified. 16. In adding apparatus, the combination with main rocking bars and a plurality of the series of key-levers, a plurality of the series 115 of cam-plates carried by a plurality of the main rocking bars and means for transmitting the movement of said main rocking bars to a plurality of registering-drums, including a pivoted arm mounted on each of the afore- 120 said main rocking bars, of means for keeping said pivoted arm normally incapable of independent movement, of means for releasing said pivoted arm and of means for moving it independently of the motion it receives from 125 the said rocking bars substantially as and

for the purpose specified.

17. In adding apparatus, the combination with main rocking bars and a plurality of the series of key-levers, a plurality of the series 130 of cam-plates carried by a plurality of the main rocking bars and means for transmitting the movement of said main rocking bars to a plurality of registering-drums, including

a pivoted arm mounted on each of the aforesaid main rocking bars, of a locking device which by the action of a spring normally lies transversely of said pivoted arm, of a sliding 5 oscillatory shaft of means for sliding said shaft to displace the locking device and so release the pivoted arm and of means for oscillating said shaft to cause the independent movement of the said pivoted arm substan-10 tially as and for the purpose specified.

18. In adding apparatus, the combination with main rocking bars and a plurality of the series of key-levers, a plurality of the series of cam-plates, a plurality of the main rocking 15 bars carrying said cam-plates, a plurality of drums and means for transmitting the movement of said rocking bars to a plurality of registering-drums, including an arm pivotally mounted on each of the aforesaid main 20 rocking bars, of a locking device comprising, a spring normally maintaining said hinged plate in a position transverse of the said pivoted arm, of a sliding oscillatory shaft connected to the pivoted arm and to the locking 25 device by a contrivance which permits the said shaft to shift longitudinally without becoming operatively disconnected from the said pivoted arm and the said locking device, of means for sliding the said shaft and of 30 means for oscillating it, all substantially as and for the purpose specified.

19. In adding apparatus, the combination with main rocking bars and a plurality of the series of key-levers, a plurality of the series 35 of cam-plates carried by a plurality of the main rocking bars, a plurality of registeringdrums and means for transmitting the movement of the said rocking bars to said plurality of registering-drums, including an arm 40 pivotally mounted on each of the main rocking bars, of a locking device comprising a spring - controlled hinged plate normally maintained transverse of said pivoted arm by its spring, of a sliding oscillatory shaft con-45 nected to the said pivoted arm by a crank, of lever mechanism adapted to be actuated by the registering-drums for causing said shaft

to slide each time the record of one drum is to be transferred to another of higher denomi-50 nation, of a projection on said sliding oscillatory shaft, of a long pin on said hinged plate against which said projection strikes and so displaces the hinged plate when the sliding oscillatory shaft is moved outward by the said

55 lever mechanism, of a finger on said sliding oscillatory shaft, and of a projection adapted to be moved each time a key-lever is operated and to actuate said finger when the latter is caused to slide into the path of said projec-

60 tion, whereby said shaft is oscillated, and, through the intervention of said crank, imparts the independent movement to the hinged arm substantially as and for the purpose specified.

20. In adding apparatus, the combination with the main rocking bar and the series of key-levers, a registering device, the series of l

cam-plates carried by the main rocking bar, and means for transmitting the movement of said main rocking bar to a registering de- 70 vice, including a movable arm mounted on the aforesaid main rocking bar, of means for moving said arm independently of the movement it receives from its rocking bar and of means for limiting the extent of such inde- 75 pendent movement substantially as and for

the purpose specified.

21. In adding apparatus, the combination with the main rocking bar and the series of key-levers, a registering device, the series of 80 cam-plates carried by the main rocking bar and means for transmitting the movement of the main rocking bar to a registering device, including a movable arm mounted on the aforesaid main rocking bar, of means for mov- 85 ing said arm independently of the movement it receives from its rocking bar and of a hinged stop-piece which normally occupies a position to limit the extent of independent movement of the said pivoted arm when the 90 latter is shifted about its pivot, substantially as described.

22. In adding apparatus, the combination with the main rocking bar and the series of key-levers, a registering device, the series of 95 cam-plates carried by the main rocking bar and means for transmitting the movement of the main rocking bar to a registering device, including a movable arm mounted on the aforesaid main rocking bar, of means for mov- 100 ing said arm independently of the movement it receives from its rocking bar, of a hinged stop-piece which normally occupies a position to limit the extent of independent movement of the said movable arm, and of means 105 for displacing said stop-piece when the said arm is to be moved with the rocking bar substantially as described.

23. In adding apparatus, the combination with main and additional rocking bars and 110 the movable arm carried by the main rocking bar, the arm carried by the additional rocking bar, and the hinged stop-piece, of means whereby said hinged stop-piece is shifted from its stopping position whenever the ad- 115 ditional rocking bar is actuated substan-

tially as described.

24. In adding apparatus, the combination with main and additional rocking bars and the movable arm carried by the main rock- 120 ing bar, the arm carried by the additional rocking bar, and the hinged stop-piece; of an incline on said hinged stop-piece and of a pin on said arm of the additional rocking bar, whereby said hinged stop-piece is shifted out 125 of the path of the movable arm of the main rocking bar by said pin, each time the additional rocking bar is actuated, substantially as and for the purpose specified.

25. In adding apparatus, the combination 130 with main and additional rocking bars and the series of key-levers, a registering device, the series of cam-plates carried by the main and additional rocking bars and means for

transmitting the movement of the main rocking bar to a registering device, including a movable arm mounted on the main rocking bar, of an arm carried by the additional rock-5 ing bar and adapted to act as a stop-piece for limiting the movement of the main rocking bar, of means for moving said movable arm independently of the movement it receives from its rocking bar, of a hinged stop-piece to for limiting the extent of such independent movement, of a locking-plate normally occupying a position transverse of and between the said movable and fixed arms, of means for displacing said hinged stop-piece when 15 the additional rocking bar is actuated and of means for withdrawing said locking-plate from its position between the fixed and movable arms, so that sufficient space will be left between these arms to permit of the inde-20 pendent movement of the movable arm when such independent movement has to be performed in addition to the movement due to the rocking of the main rocking bar.

26. In adding apparatus, the combination with main rocking bar and the pivoted arm carried by the main rocking bar, the locking device therefor and the sliding oscillatory shaft for actuating said pivoted arm and locking device, of a spring for returning said shaft to its normal position and of means for retarding said return of the shaft substantially as and for the purpose specified.

27. In adding apparatus, the combination

with the pivoted arm, the locking device and the sliding oscillatory shaft, of a spring for 35 returning said shaft longitudinally to its normal position, of a pivoted finger adapted to engage with and hold the said shaft when the latter slides into a position to be oscillated and of means for releasing said finger and 40 liberating the shaft substantially as and for

the purpose specified. 28. In adding apparatus, the combination with the pivoted arm, the locking device and the sliding oscillatory shaft, of a spring for 45 returning said shaft longitudinally to its normal position, of a pivoted finger adapted to engage with a notch in said shaft when the latter slides into a position to be oscillated, of a spring normally tending to keep said fin- 50 ger in engagement with said notch and of a rocking frame having a transverse bar extending above all the key-levers and adapted to normally keep said finger out of engagement with the notch in the shaft and to permit such 55 engagement to take place each time a keylever is operated and the frame raised, substantially as and for the purpose specified.

In testimony whereof I have hereunto set my hand, in presence of two subscribing wit- 6ō nesses, this 15th day of September, 1899.

EDWARD MORIARTY.

Witnesses:
J. Collins,
FRED C. HARRIS.