Patented June 20, 1899.

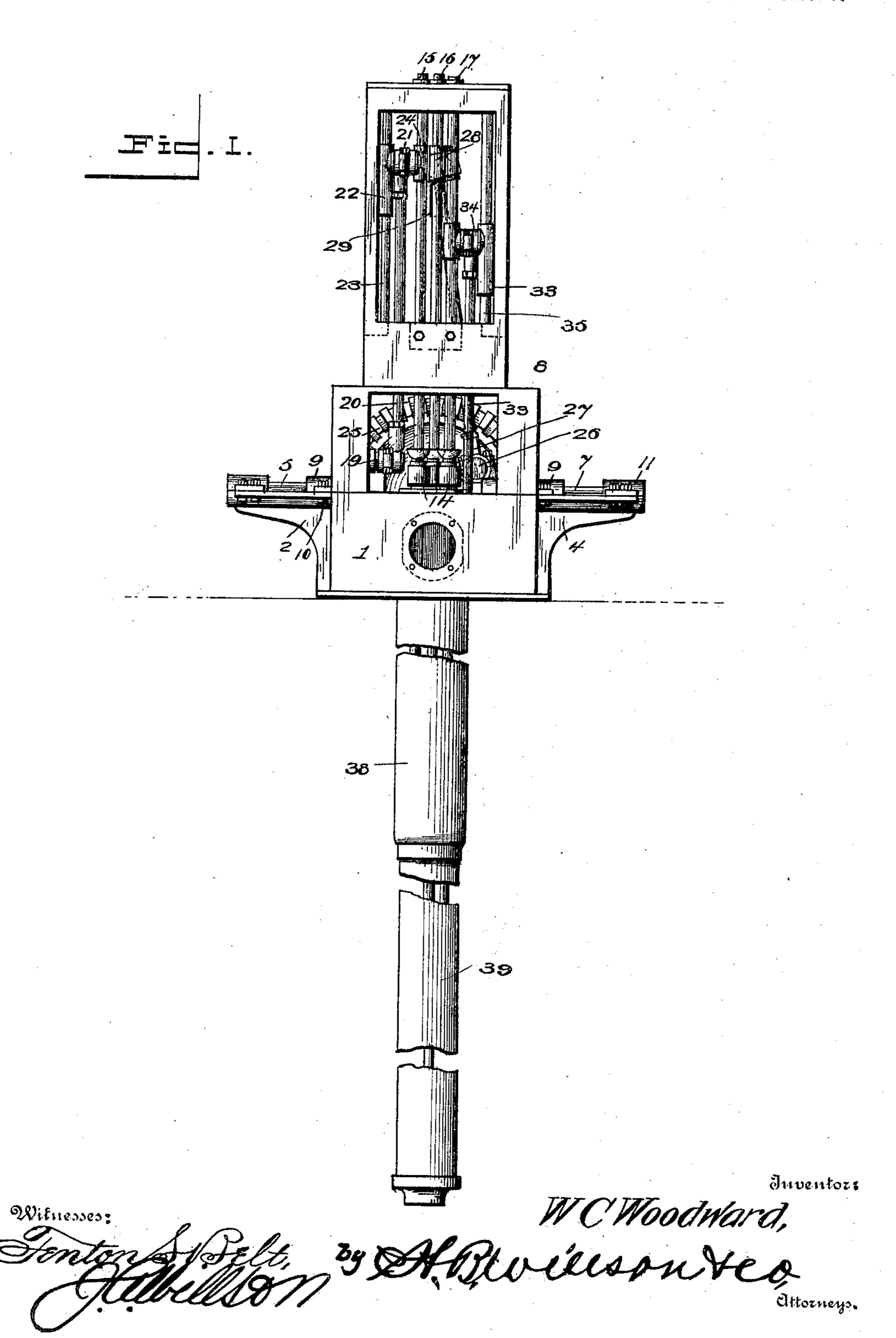
## W. C. WOODWARD.

### TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

8 Sheets-Sheet 1.



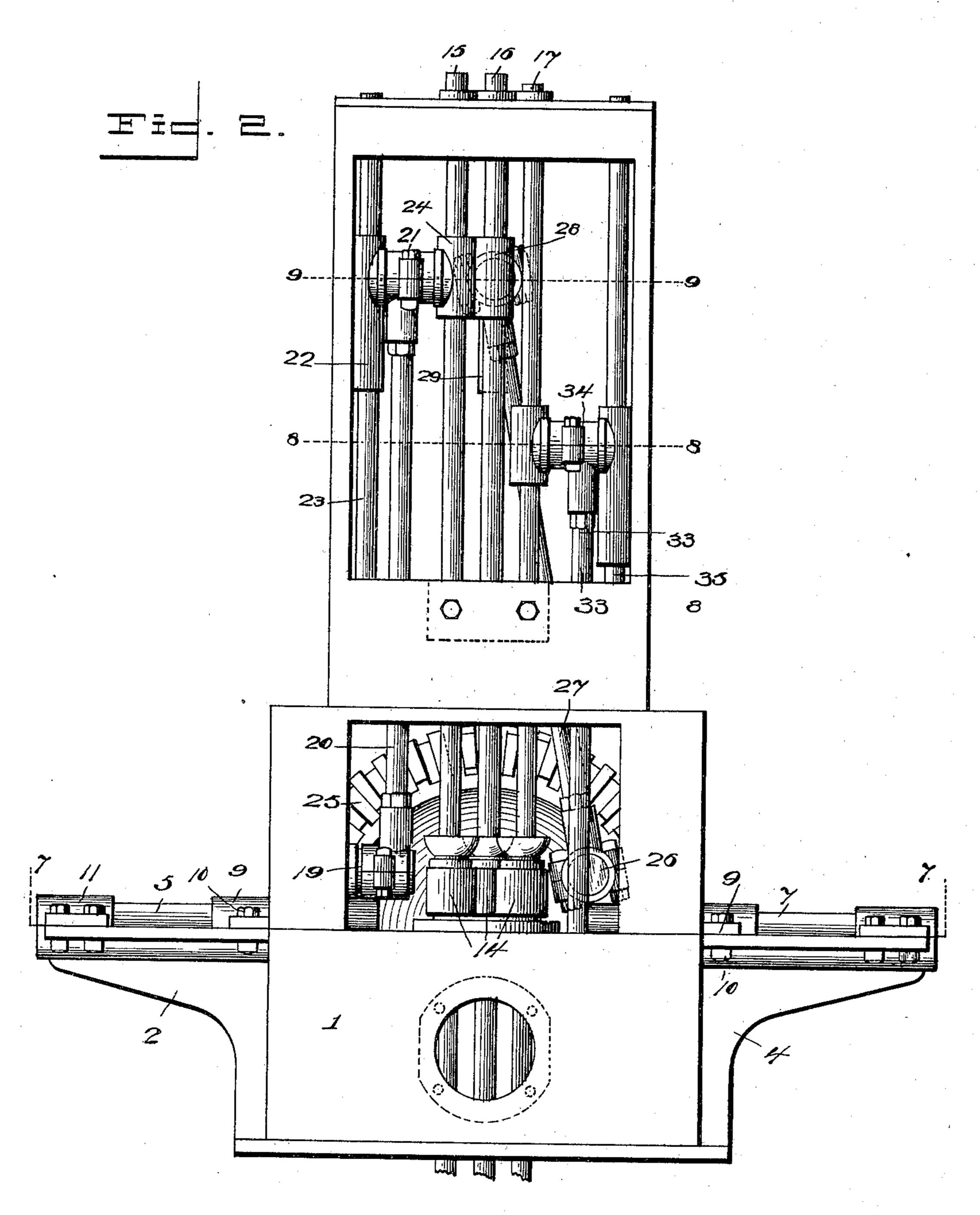
Patented June 20, 1899.

# W. C. WOODWARD. TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

8 Sheets—Sheet 2.



WCWoodward,
by Allvillson Veo.

Attorneys. Witnesses:

Patented June 20, 1899.

## W. C. WOODWARD.

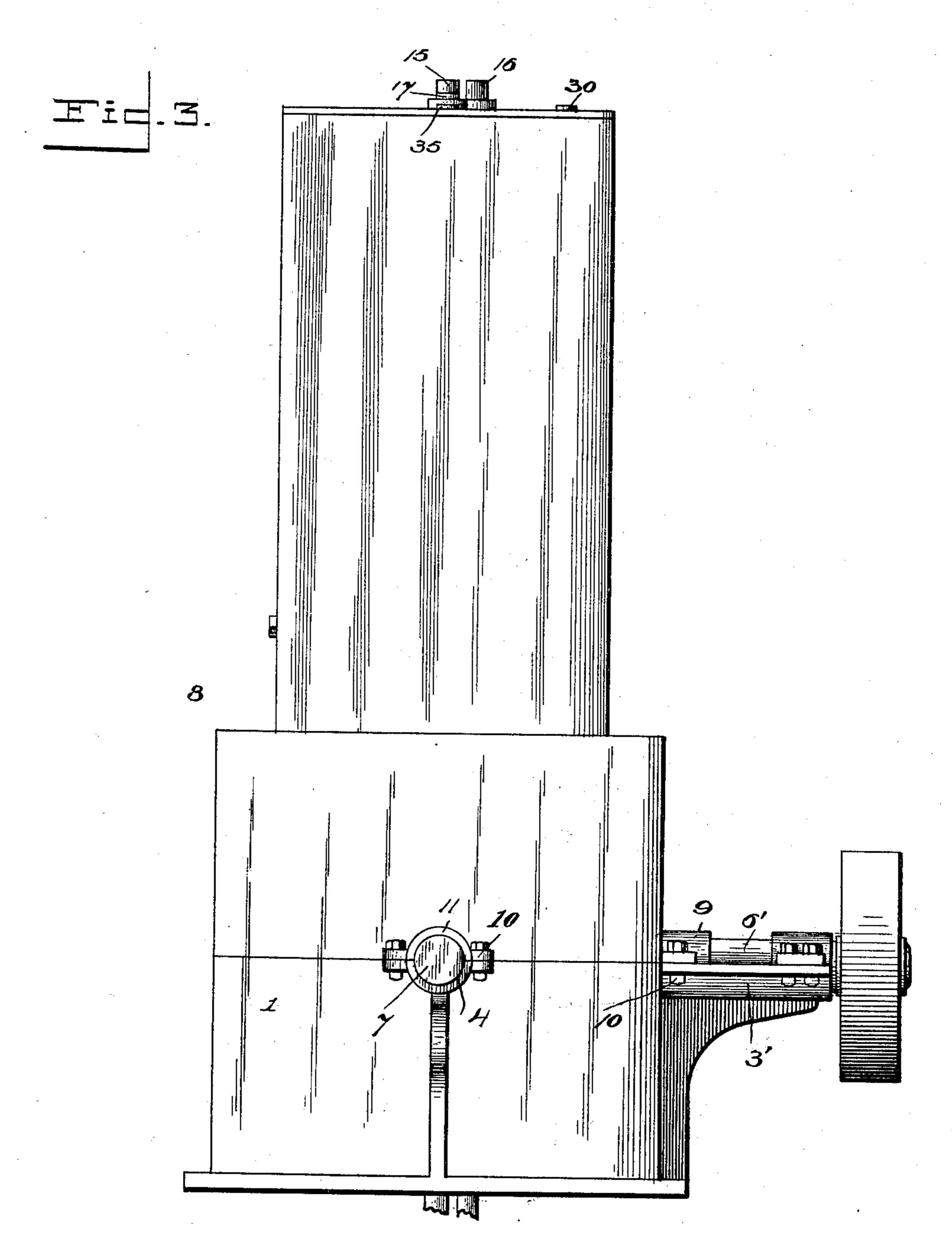
### TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

Witnesses:

8 Sheets—Sheet 3.



W.C. Woodward, ABwillsontes

# W. C. WOODWARD. TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.) 8 Sheets-Sheet 4. Inventor: W.C.Woodward, viceson Heo, Witnesses:

Patented June 20, 1899.

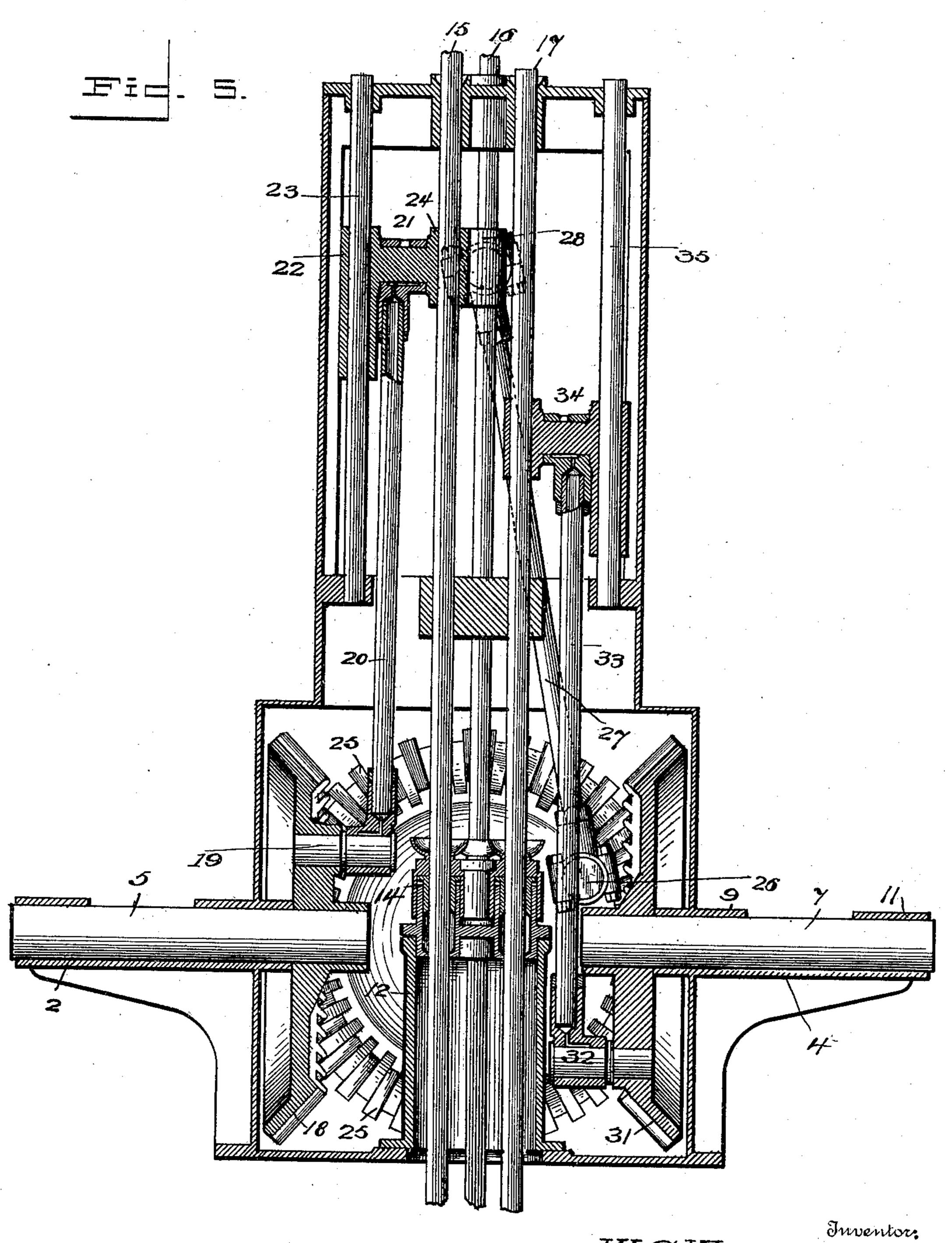
# W. C. WOODWARD.

# TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

8 Sheets-Sheet 5.



Witnesses:

M.C. Woodward, Villson Hea

Patented June 20, 1899.

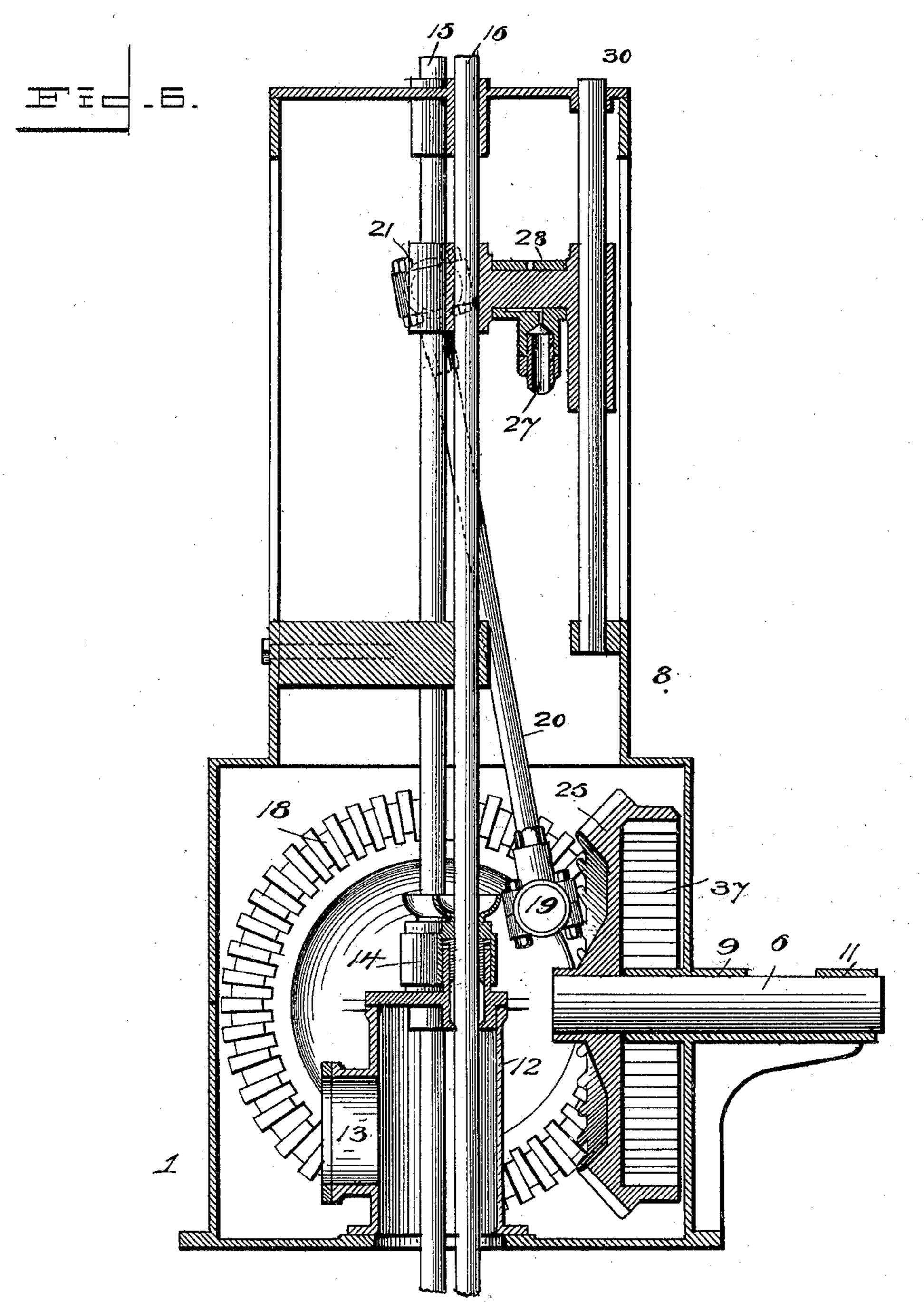
#### W. C. WOODWARD.

#### TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

8 Sheets—Sheet 6.



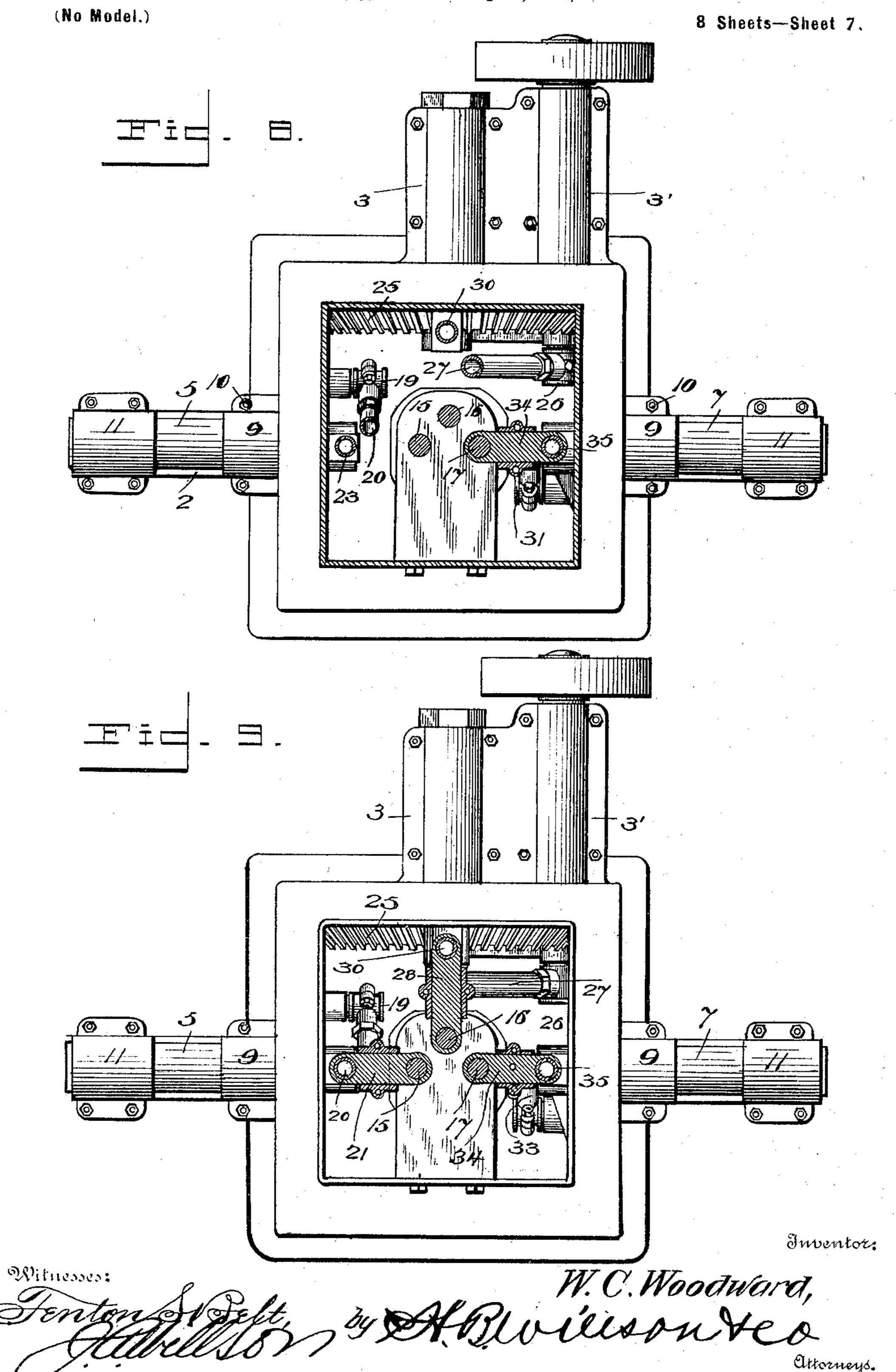
Inventor:

Witnesses:

W.C. Woodward, Bluiceson tes

# W. C. WOODWARD. TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)



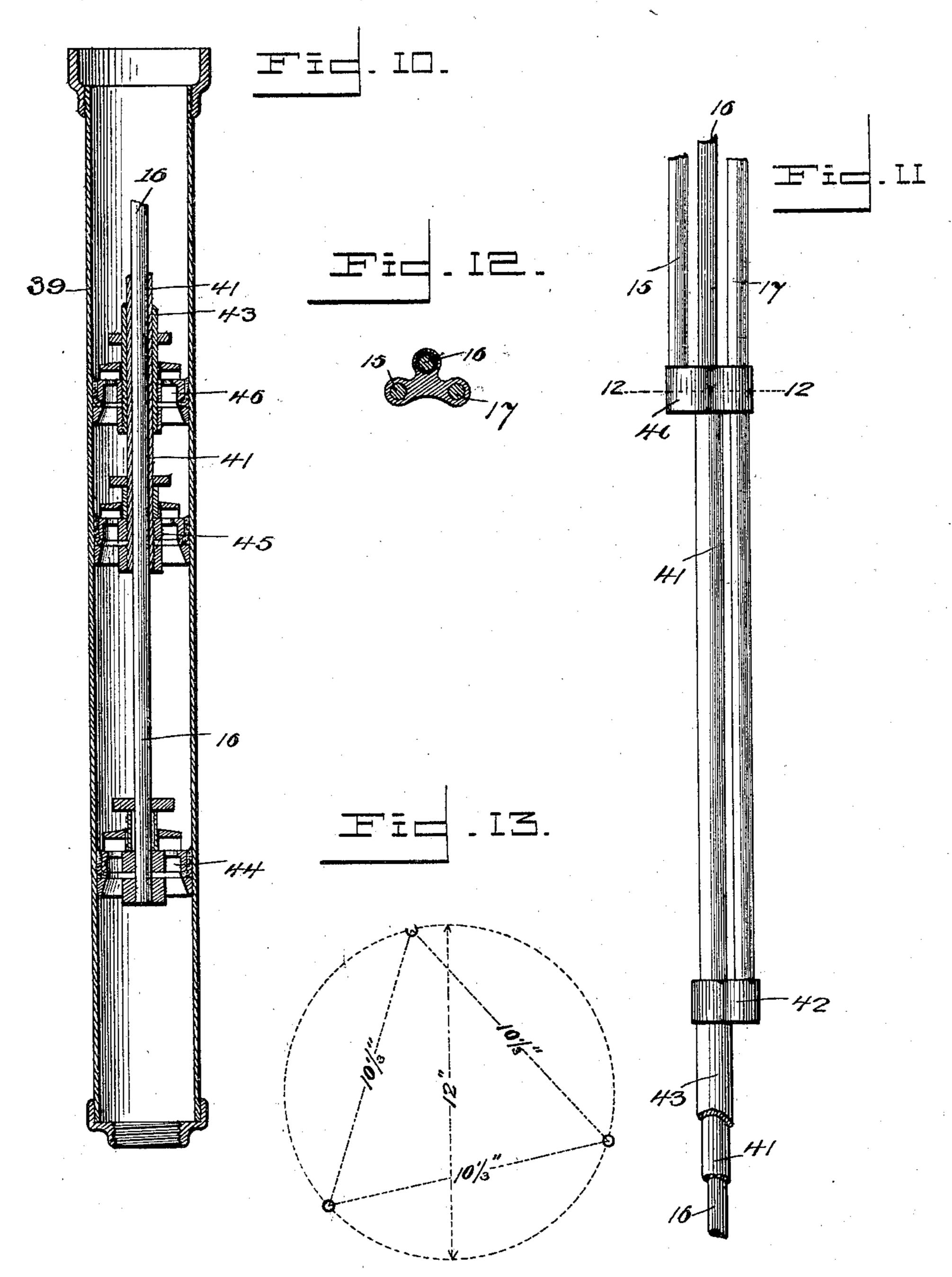
Patented June 20, 1899.

## W. C. WOODWARD. TRIPLE PISTON DEEP WELL PUMP.

(Application filed Aug. 22, 1898.)

(No Model.)

8 Sheets-Sheet 8.



Triventor.

Witnesses:

W.C. Woodward Willson Hea

# United States Patent Office.

WILLIAM CALVIN WOODWARD, OF LOS ANGELES, CALIFORNIA.

# TRIPLE-PISTON DEEP-WELL PUMP.

SPECIFICATION forming part of Letters Patent No. 627,315, dated June 20, 1899.

Application filed August 22, 1898. Serial No. 689,261. (No model.)

To all whom it may concern:

Beit known that I, WILLIAM CALVIN WOOD-WARD, a citizen of the United States, residing at Los Angeles, in the county of Los Angeles and State of California, have invented certain new and useful Improvements in Triple-Piston Deep-Well Pumps; and I do declare the following to be a full, clear, and exact description of the invention, such as will enable others skilled in the art to which it appertains to make and use the same.

My invention relates to an improved triplepiston pump for deep wells; and the object is
to provide a pump of this character whereby
by my improved construction the efficiency
will be increased by dividing the load in such
a manner that with the same expenditure of
power a greater water capacity will be attained in a given period of time than has heretofore been the case in pumps employed for
this purpose.

To this end the novelty consists in the construction, combination, and arrangement of the device, as will be hereinafter more fully described, and particularly pointed out in the claims.

The accompanying drawings show my invention in the best form now known to me; but many changes in the details might be made within the skill of a good mechanic without departing from the spirit of my invention as set forth in the claims at the end of this specification.

The same reference characters indicate the 35 same parts of the invention.

Figure 1 is a front elevation of my improved pump. Fig. 2 is an enlarged side elevation of the pump-head. Fig. 3 is a similar view of the same, taken at a right angle to that 40 shown in Fig. 2. Fig. 4 is a top plan view of the pump-head. Fig. 5 is an enlarged vertical section of the same. Fig. 6 is a similar view taken at a right angle to that shown in Fig. 5. Fig. 7 is a horizontal section on the 45 line 77 of Fig. 2. Fig. 8 is a similar view taken on the line 88 of Fig. 2. Fig. 9 is a similar view taken on the line 9 9 of the same figure. Fig. 10 is an enlarged vertical section of the pump barrel or cylinder. Fig. 11 is a 50 similar view of the triple-piston rods. Fig. 12 is a cross-section on the line 12 12 of Fig. 11.

Fig. 13 is a diagrammatic view illustrating the relative positions of the triple pistons.

1 represents the pump-head, which is recatangular in form and is provided on three 55 sides with the horizontal radiating bearing-brackets 2, 3, 3', and 4, in which are journaled the shafts 5, 6, 6', and 7.

8 denotes the hood or cover for the head, and it is formed with the integral bearing- 60 caps 9 9 9, by means of which it is removably secured to the alined bearing-brackets on the head by the bolts 10 10. Independent bearing-caps 11 11 are also secured to the outer ends of the bearing-arms to afford the proper 65 protection for the radial shafts journaled therein.

12 represents the upper section of the well casing or tube, and it is provided with the lateral outlet or discharge pipe 13 and with 70 the stuffing-boxes 14 14 14, through which the piston-rods 15, 16, and 17 pass.

18 denotes a miter-gear fixed on the inner end of the shaft 5, and its face carries a wristpin 19, from which a connecting-rod 20 ex- 75 tends upwardly to a cross-head 21, one end of which is formed with a guide-sleeve 22, which encompasses a vertical guide-rod 23, fixed in the hood 8, while its other end is formed with a collar 24, which encompasses the pis- 80 ton-rod 15, to which it is adjustably secured in any suitable manner, and of course it will be understood that a rotary motion of the gear 18 will impart a vertical reciprocating movement to the piston-rod 15. A similar 85 miter-gear 25 is fixed to the inner end of the shaft 6 and meshes with the gear 18, and it is provided with a wrist-pin 26, from which a connecting-rod 27 extends to a similar crosshead 28, one end of which carries a guide- 90 sleeve 29, encompassing the guide-rod 30, while its opposite end is fixed to the pistonrod 16, as in the first instance. A corresponding miter-gear 31 is fixed to the inner end of the shaft 7 and meshes with the gear 25, and 95 from its wrist-pin 32 a connecting-rod 33 extends to the cross-head 34, which, as in the previous instances, has one end encompassing the guide-rod 35 and its other end secured to the piston-rod 7.

The back face of the miter-gear 25 is formed with an annular internal spur-gear 36, which

meshes with a pinion 37 on the shaft 6', journaled in the bearing-bracket 3' and by means of which the three miter-gears 18, 25, and 31 are driven to simultaneously reciprocate the 5 piston-rods 15, 16, and 17. These piston-rods extend in the order shown in Fig. 12 through the well-tubing 38, holding an approximately triangular position with relation to each other, but are brought together on a common axis 10 at the pump-cylinder 39 in a manner to be

now explained.

The lower end of the piston-rod 15 terminates in a guide-yoke 40, to which is fixed a depending sleeve 41, through which the pis-15 ton-rod 16 passes, and also with a parallel guide-orifice, through which the piston-rod 17 extends. The lower end of the piston-rod 17 terminates in a guide-yoke 42, to which is fixed a depending sleeve 43, which encom-20 passes the sleeve 41 and the piston-rod 16, and to the lower end of the rod 16 the piston 44 is fixed, while the piston 45 is fixed to the end of the sleeve 41, which is a continuation of the piston-rod 15, and the piston 46 is fixed 25 to the end of the sleeve 43, which is a continuation of the piston-rod 17. These pistons 44, 45, and 46 are all similar in construction, and each is provided with a suitable valve, which lifts as the piston descends and falls 30 as the piston rises in the manner common to this class of pump-pistons.

By referring to the diagram Fig. 13 the relative course of travel of the pistons will be understood. The circumference of the cir-35 cle represents the travel of the wrist pins or cranks, which are set at one hundred and twenty degrees apart, which would be about ten and one-third  $(10\frac{1}{3})$  inches from center to

center on a twelve (12) inch stroke.

With a single-piston pump of the same stroke the column of water will be raised

twelve (12) inches each revolution, and with a double-piston pump the column of water will be raised twenty-four (24) inches, and which is the present raising limit; but by my 45 arrangement of the triple pistons one revolution will raise the column of water a height equivalent to three (3) times the distance between the centers of the crank-pins,  $(10\frac{1}{3} \times 3 =$ 31,) which is thirty-one (31) inches, a gain of 50 seven inches in the lift at each revolution over the best effective work of the doublepiston pump under the same conditions.

Having thus described my invention, what I claim as new and useful, and desire to secure 55 by Letters Patent of the United States, is—

1. In a lifting-pump, the combination with a single pump-cylinder, a triple set of valved pistons axially mounted within said cylinder, of a triple set of intermeshing miter-gears, 60 means for operating said gears and a series of piston and connecting rods independently connecting each gear with its respective piston, substantially as and for the purpose set forth.

2. In a lifting-pump, the combination of a single pump-cylinder, a triple set of valved pistons axially mounted within said cylinder and a corresponding triple set of intermeshing gear-wheels, independently connected to 7c their respective pistons, and means for simultaneously rotating said gear-wheels, so as to independently reciprocate said pistons, substantially as and for the purpose set forth.

In testimouy whereof I have hereunto set 75 my hand in presence of two subscribing wit-

nesses.

WILLIAM CALVIN WOODWARD.

Witnesses: M. H. WOODWARD, LEONARD JONES.