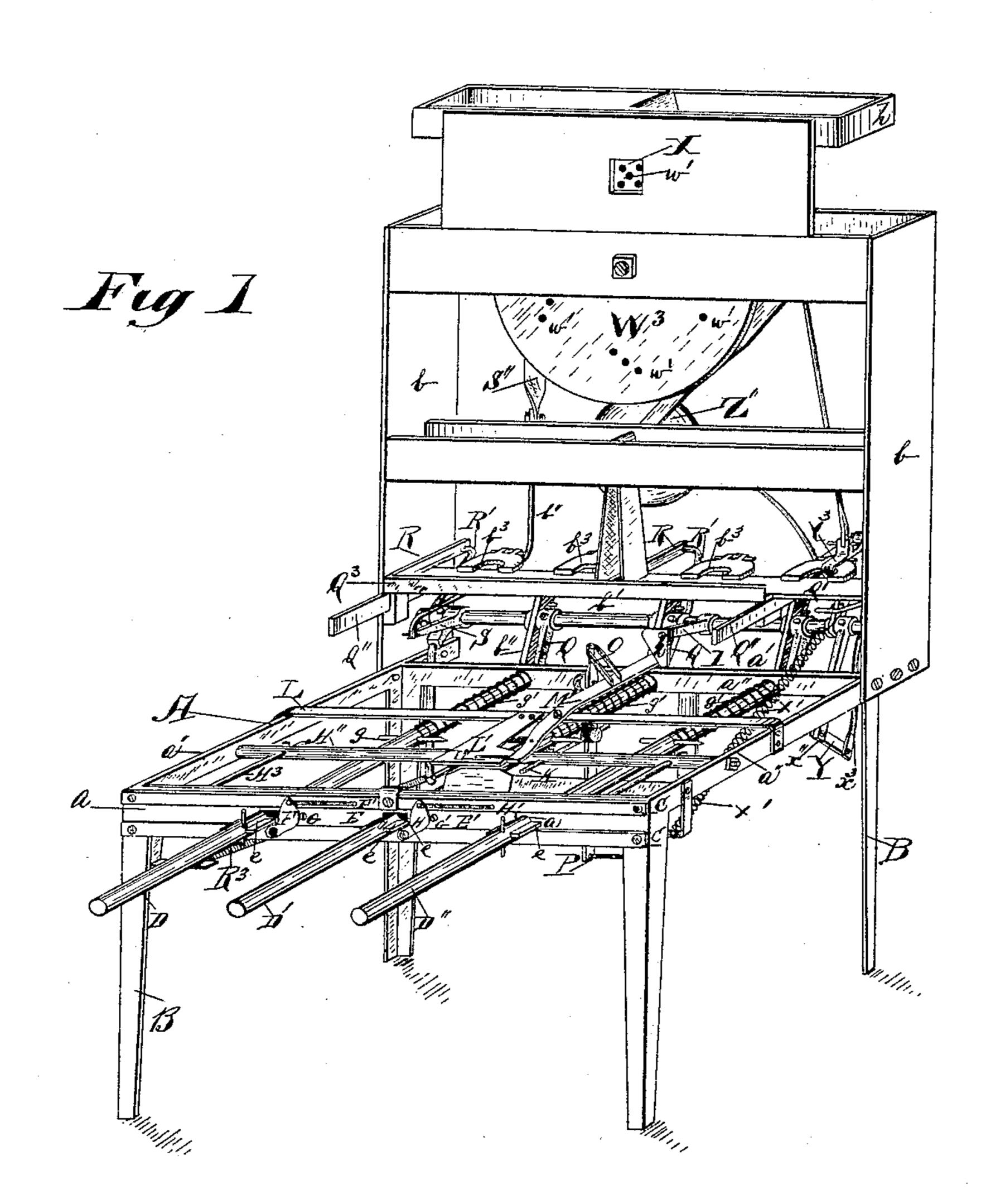
Patented July 5, 1898.

G. A. CLINE. AUTOMATIC VENDING MACHINE.

(No Model.)

(Application filed Jan. 5, 1897.)

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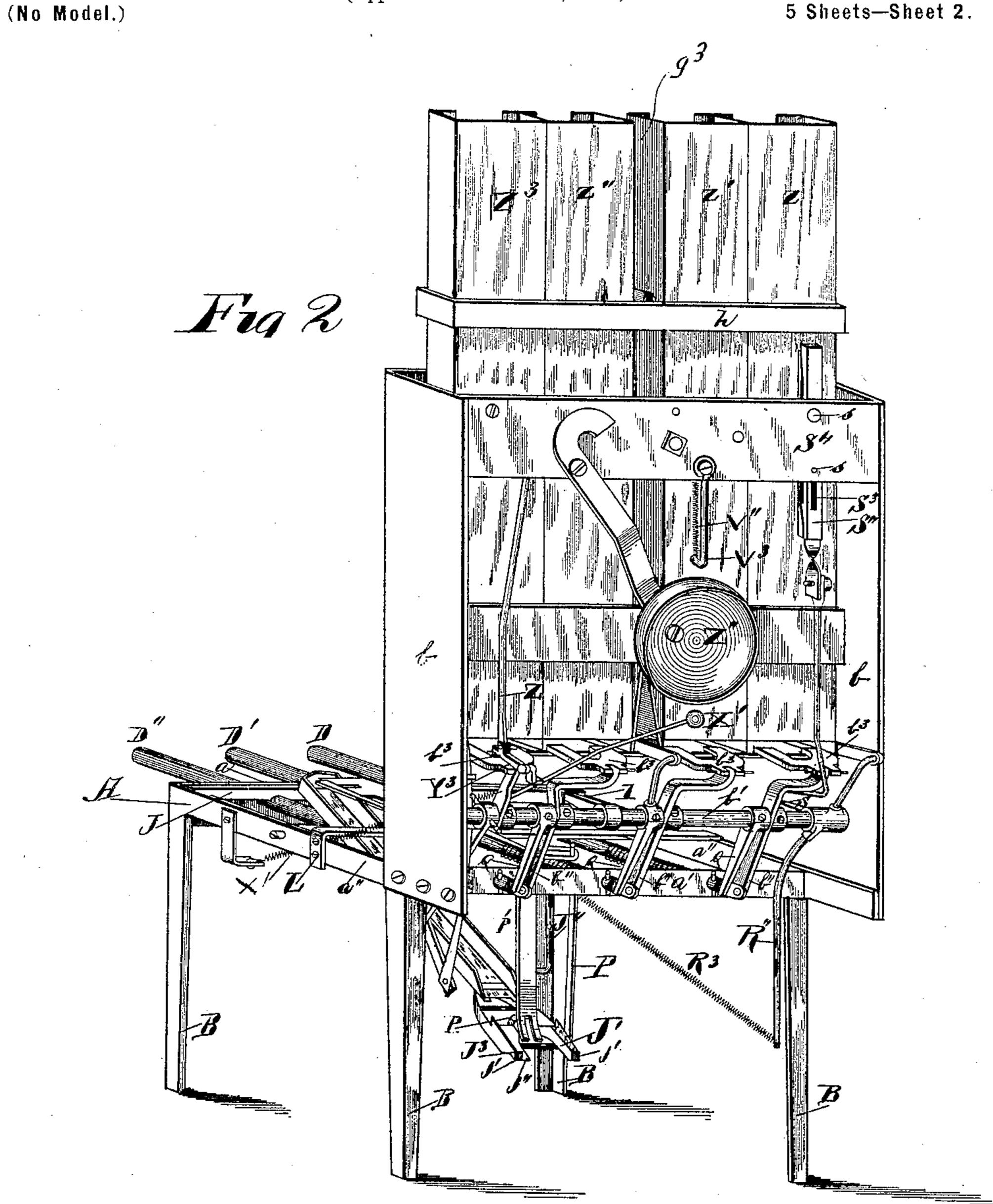
G. a. Claire by Cottles his attorney

Patented July 5, 1898.

G. A. CLINE. AUTOMATIC VENDING MACHINE.

(Application filed Jan: 5, 1897.)

5 Sheets—Sheet 2.



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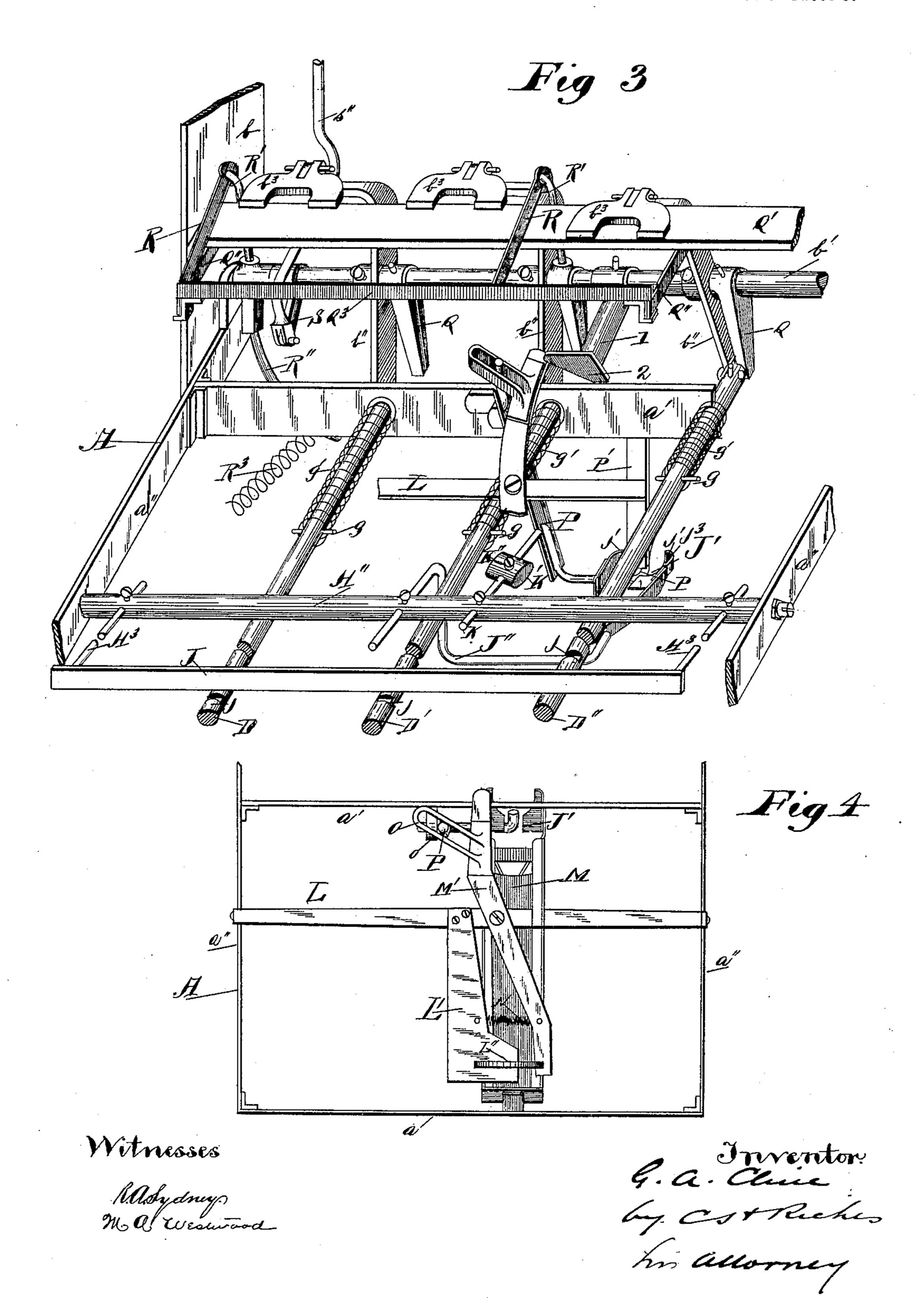
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G. A. CLINE. AUTOMATIC VENDING MACHINE.

(No Model.)

(Application filed Jan. 5, 1897.)

5 Sheets—Sheet 3.



THE NORRIS PETERS CO., PHOTO-LITHO., WASHINGTON, D. C.

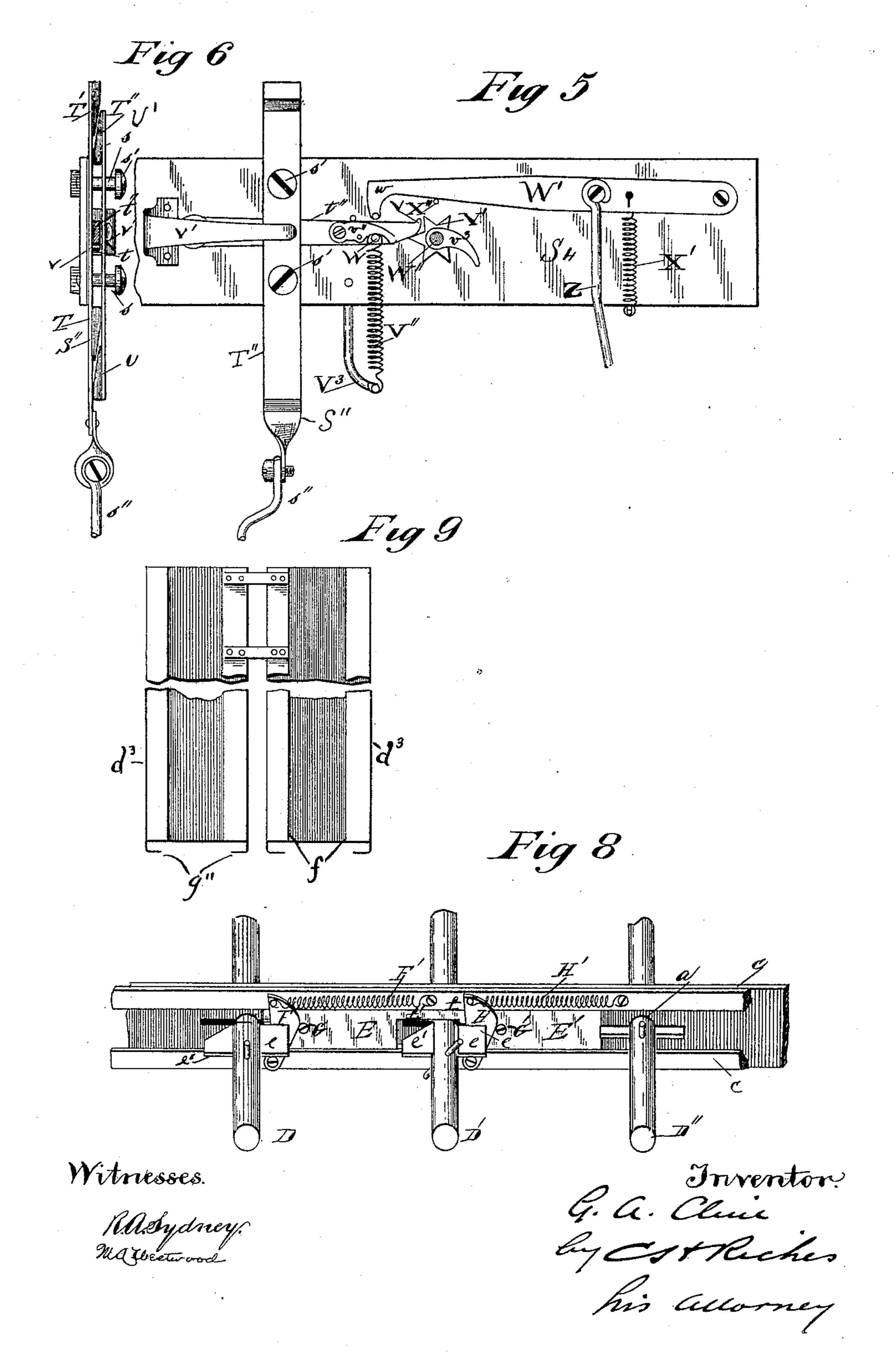
G. A. CLINE.

AUTOMATIC VENDING MACHINE.

(No Model.)

(Application filed Jan. 5, 1897.)

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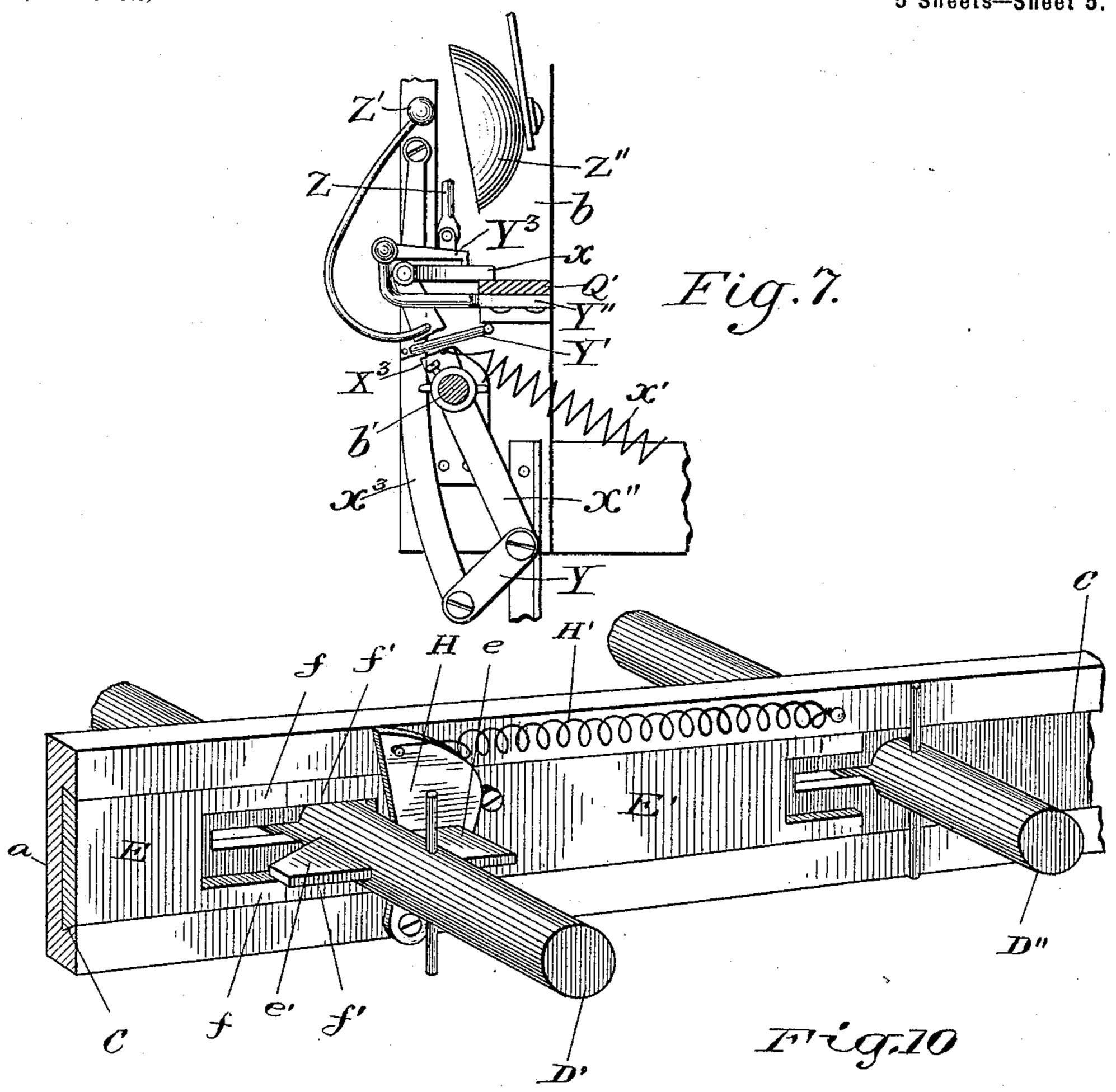


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G. A. CLINE. AUTOMATIC VENDING MACHINE.

(No Model.) (Application filed Jan. 5, 1897.)

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United States Patent Office.

GEORGE A. CLINE, OF TORONTO, CANADA.

AUTOMATIC VENDING-MACHINE.

SPECIFICATION forming part of Letters Patent No. 606,802, dated July 5, 1898.

Application filed January 5, 1897. Serial No. 618,097. (No model.)

To all whom it may concern:

Be it known that I, George Adolphus Cline, of the city of Toronto, in the county of York, in the Province of Ontario, Canada, 5 have invented certain new and useful Improvements in Automatic Vending-Machines; and I hereby declare that the following is a full, clear, and exact description of the same.

This invention relates to an automatic yend-10 ing-machine having a plurality of magazines, the goods from each magazine being discharged by a delivery mechanism primarily operated by a push-rod or plunger, the pushrods or plungers corresponding in number 15 with the number of the magazines, except where a prize-package magazine has been added to the machine; and the objects of the invention are to provide the machine with a single coin-receiving slot and chute and a 20 weighing apparatus to receive and weigh each operating coin or substance inserted into the machine and capable of being set to operate the plunger-releasing mechanism only on the insertion of a coin of the exact size and weight 25 of the coin which is required to operate the machine, to provide the plungers with a locking mechanism which will permit of the operation of only one push-rod or plunger at a single operation of the machine, to provide 30 the coin-receiving slot with a lever which will on each insertion of a coin or other substance through the coin-receiving slot cause the discharge into the coin-magazine of all coins or other substances deposited in the weighing 35 apparatus through the coin-receiving slot, to provide each plunger with a delivery mechanism to cause the delivery of one package on each complete operation of the machine from its respective magazine, to provide the 40 machine with a revoluble disk and mechanism operated by the stoppage of the revoluble disk in a predetermined position, to deliver a package from the prize-package magazine, to arrange the delivery mechanism of the 45 prize-package magazine with a gong-sounding apparatus, and to so construct the magazines that they can be removed from the machine for charging purposes, the whole device being

In the drawings, Figure 1 is a perspective | view of the front of the operating mechan-

hereinafter more fully set forth, and more par-

50 ticularly pointed out in the claims.

ism with the magazines omitted. Fig. 2 is a view of the rear of the operating mechanism. Fig. 3 is a perspective view of the primary 55 delivery mechanism and coin-testing apparatus. Fig. 4 is a plan view of the coin-testing apparatus. Fig. 5 is a front elevation of the mechanism for spinning the revoluble disk and operating the prize-delivering mechan- 60 ism. Fig. 6 is a side elevation of the mechanism for spinning the revoluble disk. Fig. 7 is a side elevation of the secondary delivery mechanism. Fig. 8 is a perspective view of the locking mechanism employed to prevent 65 the simultaneous operation of more than one plunger. Fig. 9 is a perspective view of a section of two magazines. Fig. 10 is an enlarged detail view of one of the plungers and its corresponding locking slide-sections.

Like letters and numerals of reference refer to like parts throughout the specification and drawings.

A represents the lower frame. This frame A consists of a front a, a back a', and two 75 sides a'' a'', connecting together the ends of the front a and back a'. The sides a'' a'' project beyond the back a' for the purpose of receiving the supports for part of the operating mechanism. The frame A is supsequently a sufficient length to keep the lower part of the operating mechanism entirely free from contact with the base of the machine.

C C represent two dovetailed guides con- 85 nected to the front side of the front a and extending from end to end thereof.

D D' D" represent the push-rods or plungers. Each of these plungers moves through holes d d', formed horizontally opposite each go other in the front a and back a', respectively. The holes d d' correspond in size and shape with the size and shape of the push-rods or plungers in order that no side play will be permitted on the part of the plunger. For 95 certain purposes hereinafter mentioned I provide one side of the push-rod with a lug or projection e, which forms a shoulder, behind which the latch and locking-slide are adapted to move contiguous to the outer side 100 of the front a, while I provide the other side of the push-rod with a cam e', the lesser width of the cam being contiguous to the outer side of the front a and the greater width of the

cam being remote from the outer side of the front.

E E' represent two locking-slides working in the dovetailed guides C.C. The locking-5 slide E is located between the push-rods D D', while the locking-slide E' is located between the push-rods D'D". One end of the locking-slide E normally rests against the narrow part of the cam e' in the push-rod D', 10 while the opposite end of the slide E is slightly remote from the adjacent side of the pushrod D.

Pivoted to the outer side of the front a is the lower end of a latch F, the face of which 15 is close to the lug e.

F' represents a spring connected to the upper end of the latch F and to the front a be-

tween the push-rods D D'.

G represents a pin fitted to the locking-20 slide E against the back of the latch F. By pressing inward the push-rod D' the cam e'pushes the locking-slide E toward the pushrod D. The locking-slide E instantly presses the latch F behind the lug e and holds it in 25 that position until the return of the push-rod D' to its normal position. When the pushrod D' has returned to its normal position, the spring F' withdraws the latch F from behind the lug e and simultaneously returns 30 the locking-slide into engagement with the cam e'. The locking-slide E' is provided with a latch II, a return-spring II', and pin G'.

Each of the locking-slides rests normally against the cam side of the adjacent plunger | 35 to its right, while the latches carried by the locking-slides are each arranged to pass behind the respective lugs of the plungers to the left of the operated push-rod in order that when one push-rod is operated the latches 40 of the slides to the left of the operated pushrod will quickly pass behind their respective lugs, while the locking-slides to the right of the operated push-rod will remain engaged with their respective cams, thus effectually 45 locking the remaining push-rods in their normal positions until the return of the operated push-rod to its normal position. In the event of an attempt being made to simultaneously operate two or more push-rods the 50 locking-slides to the right of the dextermost of the operated push-rods will be caused to bind against the cam sides of their respective plungers, while the latches of the lock-

ing-slides to the left of the dextermost oper-55 ated push-rod will pass behind their respective lugs, effectually locking the plungers in their normal position.

The locking-slides E are provided with rearwardly-extending arms ff, one located above 60 and the other below the push-rod D', while the locking-slide E' is provided with two forwardly-extending arms f' f', one located above and the other below the said push-rod D'. The ends of the arms ff and f'f' abut | 65 against each other in order that the slides when operated by the push-rod D" will move

does not cause the movement of either of the

locking-slides E or E'.

To explain the operation of the locking- 70 slides, we will assume that the push-rod D" has been operated. The operation of this push-rod D" causes the cam e' to move the locking-slide E' toward the push-rod D' and causes the latch H to pass behind the lug e of 75 the said push-rod, and simultaneously causes the movement of the locking-slide E toward the push-rod D and the latch F to pass behind the lug e in the said push-rod. The latches behind the lugs of the push-rods D 80 D' lock the said push-rods and prevent their operation until the return movement of the push-rod D" has been completed, when the locking-slides automatically return to their normal position. To explain the second move- 85 ment of the locking-slides, we will assume that the push-rod D has been operated. The operation of this push-rod prevents the entry of the latch F behind its luge, and thus prevents the movement of the locking-slides E 90 E', causing the locking-slides to remain in engagement with the cams e'e' of their respective push-rods and by this means preventing their operation. By operating the push-rod D' the locking-slide E is moved to- 95 ward the push-rod D and the latch F is pressed behind the lug e of the said push-rod. The operation of the push-rod D' prevents any sliding movement on the part of the lockingslide E', causing the said locking-slide to re- 100. main in engagement with the cam e' of the push-rod D".

Passing through each of the push-rods, between the front \bar{a} and back a', is a pin g, and coiled on each of the push-rods, between the 105 pin g and the inner side of the back a', is a spring g' to assist in returning the push-rods to their normal position after having been operated. Journaled in the sides a'' a'' of the frame A is a rock-shaft H", and extending for- 110 wardly from each end of the rock-shaft II" is an arm H³. Connected to the front end of the arm H³ is a locking-bar J. The top of each of the push-rods, on the inner side of the front a, is provided with a groove j, into which is 115 adapted to enter the locking-bar J, which nor-

mally rests in the groove j.

J' represents a scale-pan connected to the rock-shaft H" by a depending arm J". The scale - pan J' consists of a piece of forked 120 spring metal, each fork j' having an inwardlyextending flange j'' to support the operatingcoin or other substance inserted through the coin-receiving slot. Each forked end j' is also provided with a small inwardly-extending 125 vertical flange J³ to hold the operating-coin. All coins or other substances of a less diameter than the width of the space between the flanges J³ will pass between the said flanges into the coin-magazine.

K represents an arm extending horizontally from the rock-shaft H". Mounted on the arm K is a balance K', fitted with a set-screw K" unitedly. The push-rod D when operated I for the purpose of holding the balance in any

130

set position. The balance can be moved on the arm K", so that the weight required to upset the locking-bar J can be regulated to the infinitesimal part of an ounce. Assume, 5 for example, that the weight of a new onecent piece is required to operate the machine. The balance K' is set so that the weight of the once-cent piece in the scale-pan will raise the locking-bar Jout of the grooves j and permit 10 of the operation of any one of the push-rods. If the weight of the coin or other substance in the scale-pan is less than the weight of the one-cent piece, it will fail to raise the lockingbar out of the said grooves, and consequently 15 the push-rods will remain locked.

L represents a bracket connected to the sides a" a" between the rock-shaft H" and the back a'. Projecting forwardly from the bracket L is a plate L'. Formed inwardly 20 from one edge of the plate L' is a slot L", the length of which is less than the diameter of

the operating-coin.

M represents the coin-chute, the upper end of which is located directly below the slot \mathbf{L}'' 25 and the lower end of which discharges into the scale-pan J'. The slot L'' is directly below a corresponding slot in the case of the vendingmachine, the length of slot being preferably the same as the diameter of the operating-30 coin.

M' represents a lever pivoted at its middle to the bracket L in close proximity to the plate L'. Connected to the lever M' and to the plate L' is a spring N, which holds the 35 front end of the lever M' close against the adjacent side of the plate L' and across the slot L" in order that the entry of the operatingcoin or any other substance through the slot L" into the machine will move the adjacent 40 end of the lever M^\prime outward from the plate L^\prime and move the opposite end of the lever M' in

the opposite direction.

O represents a slotted cam connected to the rear end of the lever M'. Projecting in-45 wardly from the inner side of the back a' is a lug o, and pivoted to the lug o is the upper end of the coin-discharging lever P. The top of the coin-discharging lever P works in the slotted cam O, so that the movement of the 50 lever M' will cause a simultaneous movement on the part of the lever P. The lower end of the lever P extends below the scale-pan J' and projects upwardly through the same. The normal position of the upwardly-turned end 55 p of the lever P is at the front of the place where the operating-coin will lie when deposited in the scale-pan J'. Connected to the back a', opposite the scale-pan \mathfrak{I}' , is a depending arm p', the lower end of which is forked 60 to permit of the operation of the lever P. The purpose of the depending arm p' is to arrest the coin as it is discharged from the coinchute M and direct it into the pan J'.

The operation of this part of the invention 65 is as follows: The coin or other substance intended to operate the machine is inserted through the slot L". It descends through the

coin-chute M, and is directed by the depending arm p' into the scale-pan J'. If the coin is of the requisite weight to operate the ma- 70 chine and of sufficient size to be retained by the scale-pan, it will weigh down the pan J' and turn the rock-shaft H" to raise the locking-bar J out of the grooves j. When the locking-bar J is raised from the grooves j, any 75 one of the push-rods or plungers can be pushed inward. The passage of the coin through the slot L" moves the lever M' and causes the slotted cam O to operate the lever P and discharge all substances from the scale-pan.

We will assume that a body of the same size but of less weight than that of the requisite coin has been inserted through the slot L". This body, descending into the scale-pan J', will fail to weigh down the scale-pan. Sup- 85 pose that a second body is afterward inserted through the slot L" to make up the necessary weight. The entrance of the second body causes the operation of the levers M' and P, which delivers the first body from the scale- 90 pan. Thus it will be seen that while a coin or body of requisite size and weight will operate the machine a coin of less than the requisite weight and size will fail to operate the machine, and on the insertion of a second 95 spurious coin or other substance to complete the necessary weight the first spurious coin inserted will be discharged from the scalepan.

Each of the push-rods or plungers projects 100 beyond the back a'. Connected to the back of the sides $a^{\prime\prime}$ $a^{\prime\prime}$ are the standards b b of the upper frame. Journaled in the standards b b, \bar{s} lightly above the sides a'' a'', is a rockshaft b'.

Loosely mounted on the rock-shaft b' are a plurality of arms b'', corresponding in number and location with the push-rods or plungers D D' D". The lower end of each of the ārms b" is hinged to its respective push-rod 110 or plunger, while hinged to the upper end of each of the arms b'' is a forked plate b^3 . Each of the plates b^3 rests on the bottom of its respective magazine and is adapted to deliver one package of goods from its maga- 115 zine on each operation of its respective pushrod. By loosely mounting the arms b^3 on the rock-shaft b' any one of the arms can be operated without causing the operation of any of the remaining arms. Thus any one of the 120 push-rods or plungers can be operated to cause the delivery of the package of goods from its respective magazine without causing the delivery of the package from any of the other magazines.

Rigidly mounted on the rock-shaft b' are a series of fingers Q, one located contiguous to each of the arms b''. The ends of each of the fingers Q bear against the ends of its respective push-rod or plunger. By pushing the 130 push-rod inwardly the inner end of the pushrod bears against its respective finger Q and turns the rock-shaft b'' in its bearings for the purpose of operating certain mechanism

hereinafter mentioned. The fingers Q, in addition to turning the rock-shaft b'', cause the immediate return of the push-rods or plungers and coacting parts to their normal posi-5 tion immediately on the release of the oper-

ating-pressure.

Mounted on and moving in conjunction with the rock-shaft b' is a forwardly-extending arm 1, and carried by the arm 1 is a cam 10 2, which is arranged to engage the adjacent end of the lever M' and move the said lever on each operation of rock-shaft. The operation of the lever M', accomplished by the cam 2, has the same effect as that accomplished 15 by the insertion of the coin through the coinreceiving slot.

Connected to the standards b b is a plate Q' of substantially the same width as the width of the packages, serving as a bottom 20 for the magazines. The plate Q' supports the forked delivery-plates b³ when at rest and in motion. Projecting outwardly from the plate Q' are two guide-rods Q", and supported on the guide-rods Q" is a sliding bar Q3, hav-25 ing a vertical flange normally resting against the front of the plate Q' to retain the bottommost package within the magazine until delivered by the action of the push-rod or plun-

30 rearwardly-extending arms R R. R R' represent two arms, each rigidly con-

nected to the rock-shaft b' and pivotally connected to its respective arm R. When the rock-shaft b' has been operated by the plun-35 ger or push-rod, the forward movement of the shaft causes the arms R' and R to move the sliding bar away from the front of the plate

ger. Connected to the sliding bar Q³ are two

Q' to allow of the operated forked deliveryplate b^3 pushing a package from its respective 40 magazine and delivering the package between the sliding bar Q³ and plate Q'. The return movement of the rock-shaft b' causes a return

movement on the part of the arms R' R and the sliding bar Q³ to its normal position 45 against the plate Q'. This sliding bar serves a double purpose: first, to prevent the discharge of any package from the magazine except on a complete operation of the machine, and, second, to replace in its normal position

50 any package partially discharged, in order that the next complete operation of the machine will not cause the discharge of more

than one package.

Rigidly connected to the rock-shaft b' is a 55 downwardly-extending arm R", and connected to the arm R" and to the frame A is a spring R³, the purpose of which is to cause the quick return of the rock-shaft and coacting parts to their normal position after having been opo erated.

S represents a forwardly-extending arm rig-

idly connected to the rock-shaft b'.

S4 represents the back of the upper frame, supporting a supplemental mechanism oper-65 ated by the action of the rock-shaft, arranged to spin a disk and deliver a package from a

supplemental magazine immediately on the stoppage of the disk at a predetermined spot.

S" represents a vertical plate slidably connected to the back S4. As shown in the draw- 70 ings, the plate S" is provided with a vertical elongated slot S³, through which pass rivets, screws, or bolts s, having enlarged heads s'. The elongated slot S³ is substantially the same width as the diameter of the bolts s, and these 75 bolts not only guide the vertical movement of the plate, but arrest the movement when completed in either direction.

s" represents a connecting-rod, one end of which is pivotally connected to the arm S, and 80 the other end of which is pivotally attached to the plate S". The rock-shaft b' during its forward motion causes the downward movement of the arm S, the connecting-rods", and

plate S".

TT' represent two cams carried by the vertical plate S", the cam T being located at or near the bottom, and the cam T' at or near the top of the plate.

T" represents a plate similar to the plate 90 S" and maintained in a position opposed to

the plate S" by the bolts s.

U U' represent two cams carried by the plate T", similar in shape and opposed to the cams TT'. Each of the cams carried by the 95 plates S"T" is an inclined plane, the plane of the cams U U' being the inverse of the cams TT', respectively. The shank of each of the bolts s is of sufficient length to allow of a lateral movement on the part of the plate T" roo equal to the depth of the opposed cams. That side of the plate T" adjacent to the plate S" is provided with a stop t, against which normally rests a dog t'', pivoted to the back S^4 . The dog t'' is provided with a tooth V, adapt- 105 ed to engage the teeth of a pinion-wheel V'.

V"represents a spring connected to the dog t" and to an arm V3, depending from the back S4. The purpose of the spring V" is to draw downward the dog $t^{\prime\prime}$ when released from the 110 stop t. The quick downward movement of the $\log t''$ rapidly revolves the pinion-wheel and its shaft. To explain this movement more fully, it will be assumed that the parts have been operated to cause the downward 115 movement of the plate S" and its cams TT'. The engagement of the cams TT' with the cams U U', respectively, causes the outward movement of the plate T", carrying with it the stop t, releasing the dog t'', and allowing 120 the spring to draw downward the said dog.

v represents a pin connected to the inner side of the plate S". The purpose of the pin v is to engage the dog t' and move it upward during the return movement of the plate S" 125 in order that the stop t may engage again the dog when the parts have been returned to their normal position. To cause the sharp return of the plate T" to its normal position after having been operated, I connect to the 130 upper part of the frame a spring v', which is adapted to bear against the front of the plate

T" and press the plate T" against the plate S". Pivoted to the dog t" is a supplemental $\log v''$, which is adapted to engage a supplemental tooth v^3 on the side face of the pinion-5 wheel V'. The toothed end of the dog v''normally rests on a pin W, connected to the side face of the dog t''. When the toothed end of the dog v'' is in engagement with the supplemental tooth v^3 of the pinion-wheel V', 10 the said tooth raises the dog off the said pin W into engagement with the short arm w of an L-shaped lever W' to raise the said lever to operate the mechanism for delivering a package from the supplemental magazine. 15 The pinion-wheel V' is rigidly mounted on a

spindle W", and rigidly mounted on the same its face dice or card spots w'.

spindle W" is a disk W3, having indicated on The front of the vending-machine is fitted 20 with an opening X, through which one spot or set of spots w' is visible at a time. The inthrust of the push-rod or plunger causes the forward rocking movement of the rockshaft, which transmits, by means of the con-25 necting-rod s'', a downwardly vertical sliding movement to the plate S' and cams TT'. The vertical movement of the cams T T' imparts a lateral or outward movement to the plate T'', carrying with it the stop t and releasing the 30 dog t'', allowing the spring V'' to draw quickly downward the said dog. As the tooth V of the dog t'' engages the teeth of the pinionwheel V' the quick downward movement of the tooth V, striking the adjacent tooth of 35 the pinion-wheel, sets the pinion-wheel V', spindle W", and disk W³ rapidly revolving. The revolution or spinning of the disk continues while the plunger or push-rod is held inthrust or until the force of its revolution 40 has been expended if the plunger or push-rod is held inthrust beyond a reasonable space of time. The usual practice is, however, to immediately release the plunger from the operating pressure and allow of its quick return 45 while the disk is rapidly spinning. The return of the plunger causes the simultaneous return of the rock-shaft and plates S" T". The return of the plate S" moves upward the $\log t''$ and causes the toothed end of the said 50 dog to engage the pinion-wheel V' and instantaneously stop its revolution. If the pinion-wheel V' has been stopped in such a position as to allow the tooth v^3 to engage and lift the dog v'', the delivery of an extra package 55 of goods will be effected from a supplemental magazine by the said dog v'' engaging and operating the lever W'. The dice or card spots on the disk W³ are so arranged in relation to the tooth v^3 that when the said tooth 60 engages the dog v'' the ace or any other predetermined spot will be opposite the opening X. The end of the lever W'opposite its short arm w is pivoted to the back S^4 , and connected to the said lever and back is a spring X'. 65 Connected to the back S^4 is a pin or stop X''

W', and the purpose of the spring X' is to cause the sharp return of the lever to the pin or stop X'' after having been operated.

Loosely mounted on the rock-shaft b' is an 7°. arm X^3 , and hinged to the top of the arm X^3 is a forked delivery-plate x, which normally rests on the top of the plate Q', immediately below the bottom of the supplemental magazine. Connected to the arm X^3 and to the 75 frame A is a spring x' for the purpose of causing the return of the arm X3 with the rockshaft b' after having been operated. Depending from the rock-shaft b' is a finger x''.

 x^3 represents a lever the upper end of which 80 is pivotally connected to the standard b, adjacent to the arm X3, the opposite end of the lever x^3 being connected to the finger x'' by a

link Y.

Y' represents a catch connected to the le- 85 ver x^3 and adapted to engage the arm X^3 during the rearward movement of the lever x^3 , to move the said arm into its rearmost position, to bring the plate x clear of the bottom magazine, to allow of the bottommost pack- 90 age in the magazine resting against the top of the plate, the said plate x being opposed to the back of the bottommost package. Extending rearwardly from the plate Q', contiguous to the arm X^3 , is a bracket Y'', to which 95 is pivotally connected a stop Y³, adapted to engage and hold the plate x when moved into its rearmost position by the action of the lever x^3 . Pivotally connected to the stop Y^3 and to the lever W' is a connecting-rod Z. 100 Connected to the arm X^3 is a hammer Z', adapted to strike a gong Z" at the back of the machine on each forward movement of the arm X³. Each operation of the rockshaft b' causes the finger x'' to move the lower 105 end of the lever x^3 rearwardly. If the arm X^3 and plate x are in their forward position, the lever x^3 will during its rearward movement draw rearwardly the arm X^3 and plate x until they have reached their rearmost po- 110 sition, where they are held by means of the stop Y³. The continued action of the rockshaft and lever x^3 has now no effect on the arm X^3 and plate x. When the tooth v^3 engages the dog v'', it raises the dog to engage 115 the end of the short arm w of the lever W'and raises the said lever, the lifting movement of which lifts the connecting-rod Z and stop Y³ sufficiently to disengage the stop from the plate x and allow the plate and arm X^3 120 to return to their normal position and deliver a package of goods from the supplemental magazine, causing at the same time the hammer Z' to strike the gong Z'. The stop Y^3 , it will thus be noticed, is released from engage- 125 ment with the plate x only when the tooth v^3 engages the dog v''. It will also be noticed that the arm X^3 and plate x are moved into their rearward position on the first operation of any plunger or push-rod after having been 130 released and remain in that position until the stop Y³ has again been raised by the acto limit the downward movement of the lever l

tion of the lever W', the rock-shaft during each operation of the plungers in the interval rocking loosely in the hub of the arm X³.

zz'z'' represent the magazines operated, re-5 spectively, by the plungers D D' D", and Z3 represents the magazine operated by the arm N³ and plate x. Each of the magazines is made preferably of sheet metal, having closed sides a^3 and back d^3 , respectively, with flanges

10 f'' overlapping the front. The bottom of each side of each of the magazines is provided with a flange g'' to support the packages, while the bottom of the front flanges f'' is cut away to provide an opening for the de-

15 livery of the packages from the magazines. Between the magazines z z'' is an opening g^3 , through which passes the spindle W" of the disk and pin. The bottom of each of the magazines is open and normally rests on the 20 plate Q', while the uppermost part of the magazine is held by an open frame h, connected to the standards b b. Each magazine is removable in order that it can be charged

with goods and afterward placed in the ma-25 chine.

Having thus fully described my invention, what I claim as new, and desire to secure by

Letters Patent, is—

1. In an automatic vending-machine the 30 combination of a magazine, a transverselygrooved push-rod adapted to deliver the goods from the magazine, a rock-shaft, a coin-testing balance consisting of a scale-pan suspended from the rock-shaft, adapted to re-35 ceive the coins after their insertion, and a

counterbalance suspended from the rockshaft to act in conjunction with the scalepan, and a locking-bar carried by the rockshaft adapted to normally engage the groove 40 of the push-rod and to be upset therefrom,

after the receipt by the scale-pan of the requisite operating-coin, substantially as specified.

2. In an automatic vending-machine a de-45 livery mechanism consisting of a push-rod, a rock-shaft, an arm loosely mounted on the rock-shaft, one end pivotally connected to the push-rod and the opposite end adapted to discharge the goods from its respective

50 magazine, and a finger carried by the rockshaft and opposed to the end of the push-rod, whereby the rock-shaft is operated by the inthrust of the push-rod, substantially as specified.

3. In an automatic vending-machine the combination of a plurality of plungers, a tilting locking-bar normally engaging all of the plungers, a coin-operated balance to actuate the tilting bar to release the plungers after

60 the insertion of the requisite coin, and a locking-slide to prevent the simultaneous operation of more than one plunger, substantially as specified.

4. In an automatic vending-machine a de-65 livery mechanism consisting of a push-rod, a rock-shaft, an arm loosely mounted on the the push-rod and the opposite end adapted to discharge the goods from its respective magazine, a finger carried by the rock-shaft and 70 opposed to the end of the push-rod, whereby the rock-shaft is operated by the inthrust of the push-rod, a tilting locking-bar normally locking the push-rod, and a coin-operated balance to actuate the locking-bar to release 75 the plunger after the insertion of the requisite coin, substantially as specified.

5. In an automatic vending-machine, a delivery mechanism consisting of a plunger, a rock-shaft, an arm loosely mounted on the 80 rock-shaft, one end of the arm hinged to the plunger, the opposite end of the arm hinged to a delivery-plate, a finger moving in conjunction with the rock-shaft, bearing against the adjacent end of the plunger, and means 85 for returning the parts to their normal position after having been operated, substantially

as specified.

6. In an automatic vending-machine, a delivery mechanism consisting of a plunger, a 90 rock-shaft, an arm loosely mounted on the rock-shaft, one end pivotally connected to the adjacent end of the plunger, a deliveryplate connected to the opposite end of the arm, a finger mounted on the rock-shaft and 95 moving in conjunction therewith, bearing against the adjacent end of the plunger, and a spring to return the parts to their normal position after having been operated, substantially as specified.

7. In an automatic vending-machine, a delivery mechanism consisting of a plunger, a rock-shaft, an arm loosely mounted on the rock-shaft, one end pivotally connected to the plunger, the opposite end pivotally con- 105 nected to a delivery-plate, a finger moving in conjunction with the rock-shaft, adapted to bear against the adjacent end of the plunger and return it to its normal position, simultaneously with the return of the rock-shaft, and 110 a spring to return the rock-shaft to its normal position after being operated, substantially as specified.

8. In an automatic vending-machine the combination of a spinning disk, having indi- 115 cated on its face a series of characters to represent a game of chance, a primary delivery mechanism arranged to spin the disk during its delivery action and to instantaneously stop the spinning of the disk by its return action, 120

substantially as specified.

9. In an automatic vending-machine the combination of a spinning disk, having indicated on its face a series of characters to represent a game of chance, a primary delivery 125 mechanism arranged to spin the disk during its delivery action and to instantaneously stop the spinning of the disk by its return action, and a secondary delivery mechanism actuated by the stoppage of the spinning disk 130 at a predetermined part of its revolution, substantially as specified.

10. In an automatic vending-machine the rock-shaft, one end pivotally connected to I combination of a spinning disk, having indi-

100

cated on its face a series of characters to represent a game of chance, a primary delivery mechanism arranged to spin the disk during its delivery action and to instantaneously 5 stop the spinning of the disk by its return action, a secondary delivery mechanism, an intermediate connection between the spinning disk and the secondary delivery mechanism, actuated by the stoppage of the disk 10 at a predetermined part of its revolution, sub-

stantially as specified.

11. In an automatic vending-machine the combination of a plurality of push-rods, a rock-shaft, a locking-bar carried by the rock-15 shaft normally engaging each of the pushrods, a scale-pan suspended from the rockshaft, consisting of two forks, each having an inwardly-extending flange to support the operating-coin, and a lever projecting between 20 the forks to clear the scale-pan of all substances on the insertion of a coin through the coin-receiving slot and on each operation of the machine, substantially as specified.

12. In an automatic vending-machine the 25 combination of a plurality of push-rods, a rock-shaft, a locking-bar carried by the rockshaft normally engaging each of the pushrods, a scale-pan suspended from the rockshaft, consisting of two forks, each having an 30 inwardly-extending flange to support the operating-coin, a lever projecting between the forks to clear the scale-pan of all substances, and a lever pivotally connected to the machine, adapted to be actuated by the inser-35 tion of a coin through the coin-receiving slot, and on each operation of the machine to actuate the first lever to clear the scale-pan, substantially as specified.

13. In an automatic vending-machine a 40 coin-testing apparatus consisting of a coinreceiving slot, a pivoted lever one end of which is contiguous to the coin-receiving slot, a cam carried by the pivoted lever, a supplemental pivoted lever one end of which is en-45 gaged by the cam, and the opposite end of which is contiguous to the scale-pan of the coin-testing balance, the pivoted levers adapt-

ed to be operated by the insertion of each coin during its passage through the coin-receiving

50 slot, substantially as specified.

14. In an automatic vending-machine a coin-testing apparatus consisting of a coinreceiving slot, a pivoted lever one end of which is contiguous to the coin-receiving slot, 55 a cam carried by the pivoted lever, a supplemental pivoted lever one end of which is engaged by the cam, and the opposite end of which is contiguous to the scale-pan of the coin-testing balance, the pivoted levers adapt-60 ed to be operated by the insertion of each coin during its passage through the coin-receiving slot, in combination with the primary delivery mechanism, released by the tilting of the scale-pan, caused by the insertion of the op-65 erating-coin, substantially as specified.

15. In an automatic vending-machine a coin-testing apparatus consisting of a pivoted

scale-pan, to receive and weigh each inserted coin, a normally-locked delivery mechanism released by the operation of the scale-pan, a 70 lever pivoted contiguous to the coin-receiving slot actuated by the insertion of the coins, and also by the action of the delivery mechanism to eject the contents of the scale-pan, substantially as specified.

16. In an automatic vending-machine a coin-testing apparatus consisting of a pivoted scale-pan, to receive and weigh each inserted coin, a normally-locked delivery mechanism released by the operation of the scale-pan, a 80 lever pivoted contiguous to the coin-receiving slot actuated by the insertion of the coins, and a cam actuated by the delivery mechanism to operate the lever during the operation of the delivery mechanism to eject the con-85 tents of the scale-pan, substantially as specified.

17. In an automatic vending-machine, the combination of a primary delivery mechanism, a secondary delivery mechanism placed 90 in position by the operation of the primary delivery mechanism, to be actuated by an intermediate mechanism controlled by the primary delivery mechanism, substantially as specified.

18. In an automatic vending-machine the combination of a primary delivery mechanism, a disk having a series of numbers or symbols indicated on one of its side faces, mounted on an easily-revoluble spindle revolved 100 by the delivery action and stopped by the return action of the primary delivery mechanism, a lug carried by the spindle, a secondary delivery mechanism, an intermediate connection for the secondary delivery mechan- 105 ism actuated by the lug on the stoppage of the revolution of the disk at a predetermined part of its revolution, substantially as specified.

19. In an automatic vending-machine the 110 combination of a primary delivery mechanism, a magazine, a permanent bottom for the magazine, an opening in the bottom of the front of the magazine for the delivery of the goods, a sliding bar opposed to the opening, 115 arms connected to the sliding bar and to the primary delivery mechanism, to allow of the primary delivery mechanism moving the sliding bar away from the opening to deliver the goods through the opening formed between 120 the bottom of the magazine and the sliding bar, and to cause the sliding bar to return the undelivered packages of goods to their original position in the magazine on the return of the primary delivery mechanism, substan- 125 tially as specified.

20. In an automatic vending-machine the combination of a plurality of plungers, a delivery mechanism operated by each plunger, a tilting locking-bar normally engaging each 130 of the plungers, a balance to actuate the locking-bar after the insertion of the requisite coin to release the plungers, and a lockingslide consisting of a series of sections each

having a spring-pressed latch adapted to engage the remaining plungers and prevent their operation while one plunger is being op-

erated, substantially as specified.

21. In an automatic vending-machine the combination of a frame, a plurality of plungers passing through the frame, guides formed in the frame contiguous to the plungers, a locking slide moving in the guides, consisting ro of a plurality of sections, one section interposed between each adjacent pair of plungers, and a spring-pressed latch carried by each section, adapted to engage the remaining plungers and prevent their operation while 15 one plunger is being operated, substantially as specified.

combination of a frame, a plurality of plungers passing through the frame, guides formed 20 in the frame contiguous to the plungers, a locking-slide moving in the guides, consisting of a plurality of sections, one section interposed between each adjacent pair of plungers, a spring-pressed latch carried by each 25 section, adapted to engage the remaining plungers and prevent their operation while one plunger is being operated, and a coin-actuated locking - bar normally engaging the plungers to prevent their operation until the 30 insertion of the requisite coin, substantially

as specified.

23. In an automatic vending-machine the combination of a frame, a plurality of plungers passing through the frame, guides formed 35 in the frame contiguous to the plungers, a locking-slide moving in the guides, consisting of a plurality of sections, one section interposed between each adjacent pair of plungers, a spring-pressed latch carried by each 40 section, adapted to engage the remaining plungers and prevent their operation while one plunger is being operated, and a cam for each plunger adapted to move the lockingslide sections to bring the spring-pressed 45 latches into engagement with their respective

plungers, substantially as specified.

24. In an automatic vending-machine the combination of a frame, a plurality of plungers passing through the frame, guides formed 50 in the frame contiguous to the plungers, a locking-slide moving in the guides, consisting of a plurality of sections, one section interposed between each adjacent pair of plungers, a spring-pressed latch carried by each 55 section, adapted to engage the remaining plungers and prevent their operation while one plunger is being operated, and a spring to return the locking-slide sections to their normal position after the return of the oper-60 ated plunger to its normal position, substantially as specified.

25. In an automatic vending-machine the combination of a frame, a plurality of plungers passing through the frame, guides formed 65 in the frame contiguous to the plungers, a ing of a plurality of sections, one section interposed between each adjacent pair of plungers, a spring-pressed latch carried by each section, adapted to engage the remaining 70 plungers and prevent their operation while one plunger is being operated, a coin-actuated locking-bar normally engaging the plungers to prevent their operation until the insertion of the requisite coin, and a spring to return 75 the locking-slide sections to their normal position after the return of the operated plunger to its normal position, substantially as specified.

26. In an automatic vending-machine the 80 combination of a frame, a plurality of plungers passing through the frame, guides formed 22. In an automatic vending-machine the | in the frame contiguous to the plungers, a locking-slide moving in the guides, consisting of a plurality of sections, one section in- 85 terposed between each adjacent pair of plungers, a spring-pressed latch carried by each section, adapted to engage the remaining plungers and prevent their operation while one plunger is being operated, a cam for each 90 plunger adapted to move the locking-slide sections to bring the spring-pressed latches into engagement with their respective plungers, and a spring to return the locking-slide sections to their normal position after the re- 95 turn of the operated plunger to its normal position, substantially as specified.

> 27. In an automatic vending-machine the combination of a plunger provided with a cam, and a groove opposed to the cam, a locking- 100 slide consisting of a plurality of movable sections arranged to move independently and in conjunction with each other, and to be actuated by the cam during the movement of the plunger, and a spring-pressed latch carried 105 by each of the locking-slide sections, adapted to enter the groove in its respective plunger,

substantially as specified.

28. In an automatic vending-machine, the combination of a primary delivery mechan- 110 ism, a secondary delivery mechanism consisting of a spring-actuated delivery-plate, a stop to hold the delivery-plate in an operative position, a pivoted lever, a connection between the stop and the pivoted lever, and an inter- 115 mediate connection between the lever and the primary delivery mechanism, substantially as specified.

29. In an automatic vending-machine, the combination of a primary delivery mechan- 120 ism, a revoluble disk, a pinion-wheel moving in conjunction with the revoluble disk, a dog engaging the teeth of the pinion-wheel, and a connection between the primary delivery mechanism and the dog, to revolve the pin- 125 ion-wheel and afterward to stop its motion, substantially as specified.

30. In an automatic vending-machine, the combination of a primary delivery mechanism, a revoluble disk, a pinion moving in con- 130 junction with a revoluble disk, a pivoted dog locking-slide moving in the guides, consist-I normally engaging the teeth of the pinion-

wheel, a spring to free the dog from engagement with the teeth of the pinion-wheel, and a movable stop operated by the primary delivery mechanism, to normally hold the piv-5 oted dog in engagement with the teeth of the pinion-wheel, and to return it to its normal position after setting the pinion-wheel in mo-

tion, substantially as specified.

31. In an automatic vending-machine the 10 combination of a primary delivery mechanism, a secondary delivery mechanism consisting of a delivery-plate, placed in position by the action of the primary delivery mechanism, a spring connected to the delivery-15 plate, a stop to hold the delivery-plate in its set position, a pivoted lever, a link connecting the pivoted lever and stop, a revoluble disk actuated by the primary delivery mechanism, adapted on its stoppage to operate the 20 pivoted lever, and release the delivery-plate from the stop, substantially as specified.

32. In an automatic vending-machine, a primary delivery mechanism, consisting of a coin-controlled plunger, a rock-shaft, an arm 25 loosely mounted on the rock-shaft, one end pivotally connected to the adjacent end of the plunger, a delivery-plate hinged to the opposite end of the arm, a finger moving in conjunction with the rock-shaft, and bearing 30 against the adjacent end of the plunger, in combination with a secondary mechanism, consisting of an arm loosely mounted on the rock-shaft, a delivery-plate hinged to the arm, a lever actuated by the rock-shaft to set the 35 delivery-plate, and a stop to temporarily hold the delivery-plate in its set position, substan-

tially as specified. 33. In an automatic vending-machine, the combination of a primary delivery mechan-40 ism, consisting of a plunger, a rock-shaft, an arm loosely mounted on the rock-shaft, one end of the arm pivotally connected to the adjacent end of the plunger, a delivery-plate hinged to the opposite end of the arm, a fin-45 ger moving in conjunction with the rockshaft, adapted to bear against the adjacent end of the plunger, and a secondary delivery mechanism consisting of an arm loosely mounted on the rock-shaft, a delivery-plate 50 hinged to the said arm, a lever actuated by the rock-shaft to set the delivery-plate in its operative position, a stop to hold the deliveryplate in its set position, a pivoted lever, a link connecting the pivoted lever and stop,

55 and a revoluble disk actuated by the primary delivery mechanism, adapted to actuate the pivoted lever to release the delivery-plate from the stop, substantially as specified.

34. In an automatic vending-machine, the 60 combination of a primary delivery mechanism, consisting of a plunger, a rock-shaft, an arm loosely mounted on the rock-shaft, one end of the arm pivotally connected to the adjacent end of the plunger, a delivery-plate 65 hinged to the opposite end of the arm, a finger moving in conjunction with the rockshaft, adapted to bear against the adjacent

end of the plunger, and a secondary delivery mechanism consisting of an arm loosely mounted on the rock-shaft, a delivery-plate 70 hinged to the said arm, a lever actuated by the rock-shaft to set the delivery-plate in its operative position, a stop to hold the deliveryplate in its set position, a pivoted lever, a link connecting the pivoted lever and stop, a rev- 75 oluble disk actuated by the primary delivery mechanism, adapted to actuate the pivoted lever to release the delivery-plate from the stop, and a spring to operate the deliveryplate of the secondary delivery mechanism 80 substantially as specified.

35. In an automatic vending-machine the combination of a delivery mechanism, a cointesting apparatus consisting of a rock-shaft, a scale-pan connected to the rock-shaft to re- 85 ceive and weigh each inserted coin, and upset the rock-shaft on the insertion of a specific coin, a lever to eject all substances contained in the scale-pan on each insertion of a coin through the coin-receiving slot, a cam oper- 90 ated by the primary delivery mechanism to actuate the said lever, and a locking-bar carried by the rock-shaft to normally engage the primary delivery mechanism, and to release the primary delivery mechanism on the in- 95 sertion of the requisite coin, substantially as

specified.

36. In an automatic vending-machine, the combination of a delivery mechanism consisting of a plurality of plungers, a rock-shaft, 100 a plurality of arms loosely mounted on the rock-shaft, one end of each arm pivotally connected to the adjacent end of its respective plunger, a delivery-plate hinged to the opposite end of each arm, a plurality of fingers 105 moving in conjunction with the rock-shaft, each finger adapted to bear against the adjacent end of its respective plunger, a forwardly-extending arm connected to the said rock-shaft, a cam carried by the said arm, a 110 coin-testing apparatus consisting of a rockshaft, a balance carried by the rock-shaft, a locking-bar connected to the rock-shaft adapted to engage each of the plungers, and a lever to remove all substances from the balance 115 actuated by the cam, substantially as specified.

37. In an automatic vending-machine, the combination of a delivery mechanism, consisting of a plurality of plungers, a rock-shaft, 120 a plurality of arms loosely mounted on the rock-shaft, one end of each of the arms pivotally connected to the adjacent end of its respective plunger, a delivery-plate hinged to the opposite end of each of the said arms, 125 a plurality of fingers, each finger adapted to engage the adjacent end of its respective plunger, a rock-shaft, a locking-bar carried by the rock-shaft normally engaging each of the said plungers, a coin-testing apparatus 130 consisting of a balance connected to the rockshaft of the locking-bar, a coin-chute to receive the coin from the coin-receiving slot, and deposit it on the scale-pan of the balance,

a pivoted lever operated by the admission of the coin, and by a cam carried by the rockshaft of the delivery mechanism, a pivoted lever, one end of which is adapted to remove 5 all substances from the scale-pan of the balance, and the opposite end of which is actuated by the pivoted lever, substantially as specified.

38. In an automatic vending-machine, the 10 combination of a delivery mechanism, consisting of a plurality of plungers, a rock-shaft, a plurality of arms loosely mounted on the rock-shaft, one end of each of the arms pivotally connected to the adjacent end of its 15 respective plunger, a delivery-plate hinged to the opposite end of each of the said arms, a plurality of fingers, each finger adapted to engage the adjacent end of its respective plunger, a rock-shaft, a locking-bar carried 20 by the rock-shaft normally engaging each of the said plungers, a coin-testing apparatus consisting of a balance connected to the rockshaft of the locking-bar, a coin-chute to receive the coin from the coin-receiving slot, 25 and deposit it on the scale-pan of the balance, a pivoted lever operated by the admission of the coin, and by a cam carried by the rockshaft of the delivery mechanism, a pivoted lever, one end of which is adapted to remove 30 all substances from the scale-pan of the balance, and the opposite end of which is engaged by a cam carried by the pivoted lever, substantially as specified.

39. In an automatic vending-machine the 35 combination of a plurality of primary delivery mechanisms, each operated by its own pushrod, a locking-bar normally engaging each of the push-rods, a rock-shaft carrying the locking-bar, and a scale-pan connected to the

40 rock-shaft, substantially as specified.

40. In an automatic vending-machine the combination of a plurality of primary delivery mechanisms, each operated by its own pushrod, a locking-bar normally engaging each of the push-rods, a rock-shaft carrying the locking-bar, a scale-pan connected to the rockshaft, a pivoted lever operated by the insertion of a coin and also by the action of the primary delivery mechanism, a cam connected 50 to the pivoted lever, a second pivoted lever, one end of which is adapted to be engaged by the said cam, and the opposite end of which is adapted to remove all substances from the scale-pan of the balance, substantially as 55 specified.

41. In an automatic vending-machine the combination of a plurality of primary delivery mechanisms, each independently operated by its own push-rod, a locking-bar nor-60 mally engaging the push-rods, a rock-shaft carrying the locking-bar, a scale-pan connected to the rock-shaft, a pivoted lever operated by the insertion of the coin, and by a cam carried by the delivery mechanism to re-65 move all substances from the scale-pan, substantially as specified.

42. In an automatic vending-machine, the

combination of a delivery mechanism, consisting of a plurality of plungers, a rock-shaft operated by each of the said plungers, a delivery- 70 plate coacting with each plunger, the magazines, a permanent bottom for the magazines, and a removable flange for the permanent bot-

tom, substantially as specified.

43. In an automatic vending-machine, the 75 combination of a primary delivery mechanism, a spindle journaled in the framework, a pinion and a disk mounted on the spindle, a pawl engaging with the teeth of the pinion, a supplemental tooth revolving with the spin- 80 dle, a trip-dog pivoted to the pawl, adapted to engage with the supplemental tooth, a pivoted lever engaging with the trip-dog, adapted to operate a secondary delivery mechanism, a primary delivery mechanism, and means 85 connected to the primary delivery mechanism to operate the pawl, substantially as specified.

44. In an automatic vending-machine, the combination of a primary delivery mechanism, a vertically-sliding plate operated by the 90 primary delivery mechanism, cams connected to the sliding plate, a laterally-moving plate, cams connected to the laterally-moving plate, engaging the cams on the sliding plate, a stop carried by the laterally-moving plate, a pawl 95 normally resting on the said stop, a spring to actuate the pawl, a pinion engaged by the said pawl, a spindle on which the pinion is rigidly mounted, a disk mounted on the said spindle, a trip-dog pivotally connected to the pawl, 100 adapted to be engaged by a tooth on the said spindle, and a stop to limit the movement of the pawl and return it to its normal position after having been actuated, substantially as

specified.

45. In an automatic vending-machine, the combination of a primary delivery mechanism, a vertically-sliding plate operated by the primary delivery mechanism, cams connected to the sliding plate, a laterally-moving plate, rrc cams connected to the laterally-moving plate, engaging the cams on the sliding plate, a stop carried by the laterally-moving plate, a pawl normally resting on the said stop, a spring to actuate the pawl, a pinion engaged by the said 115 pawl, a spindle on which the pinion is rigidly mounted, a disk mounted on the said spindle, a trip-dog pivotally connected to the pawl, adapted to be engaged by a tooth on the said spindle, a stop to limit the movement of the 120 pawl and return it to its normal position after having been actuated, a pivoted lever engaged by the trip-dog, a secondary delivery mechanism, consisting of a delivery-plate held in an operative position by a stop, a link con-125 nected to the said stop and pivoted lever, and a spring to actuate the stop, substantially as specified.

46. In an automatic vending-machine, the combination of a primary delivery mechan- 130 ism, a vertically-sliding plate operated by the primary delivery mechanism, cams connected to the sliding plate, a laterally-moving plate, cams connected to the laterally-moving plate,

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engaging the cams on the sliding plate, a stop carried by the laterally-moving plate, a pawl normally resting on the said stop, a spring to actuate the pawl, a pinion engaged by the 5 said pawl, a spindle on which the pinion is rigidly mounted, a disk mounted on the said spindle, a trip-dog pivotally connected to the pawl, adapted to be engaged by a tooth on the said spindle, a stop to limit the movement of ro the pawl and return it to its normal position after having been actuated, a pivoted lever engaged by the trip-dog, a secondary delivery mechanism, consisting of a delivery-plate held in an operative position by a stop, a link 15 connected to the said stop and pivoted lever, a spring to actuate the stop, a gong, and a hammer actuated by the return of the delivery-plate to sound the gong, substantially as specified.

20 47. In an automatic vending-machine, the combination of a primary delivery mechanism, consisting of a plunger, a rock-shaft actuated by the plunger, and a secondary delivery mechanism consisting of an arm loosely mounted on the rock-shaft, a delivery-plate

hinged to the arm, a lever pivotally connected to the framework, a lever moving in conjunction with the rock-shaft, a link connecting together the said levers, and a claw connected to the first lever to draw backward the arm 3° and delivery-plate, substantially as specified.

48. In an automatic vending-machine, the combination of a primary delivery mechanism, consisting of a plunger, a rock-shaft actuated by the plunger, and a secondary de-35 livery mechanism consisting of an arm loosely mounted on the rock-shaft, a delivery-plate hinged to the arm, a lever pivotally connected to the framework, a lever moving in conjunction with the rock-shaft, a link connecting 40 together the said levers, a claw connected to the first lever to draw backward the arm and delivery-plate, and a spring to operate the delivery-plate, substantially as specified.

Toronto, December 18, A. D. 1896.

G. A. CLINE.

In presence of— C. H. RICHES, M. A. NESTWOOD.