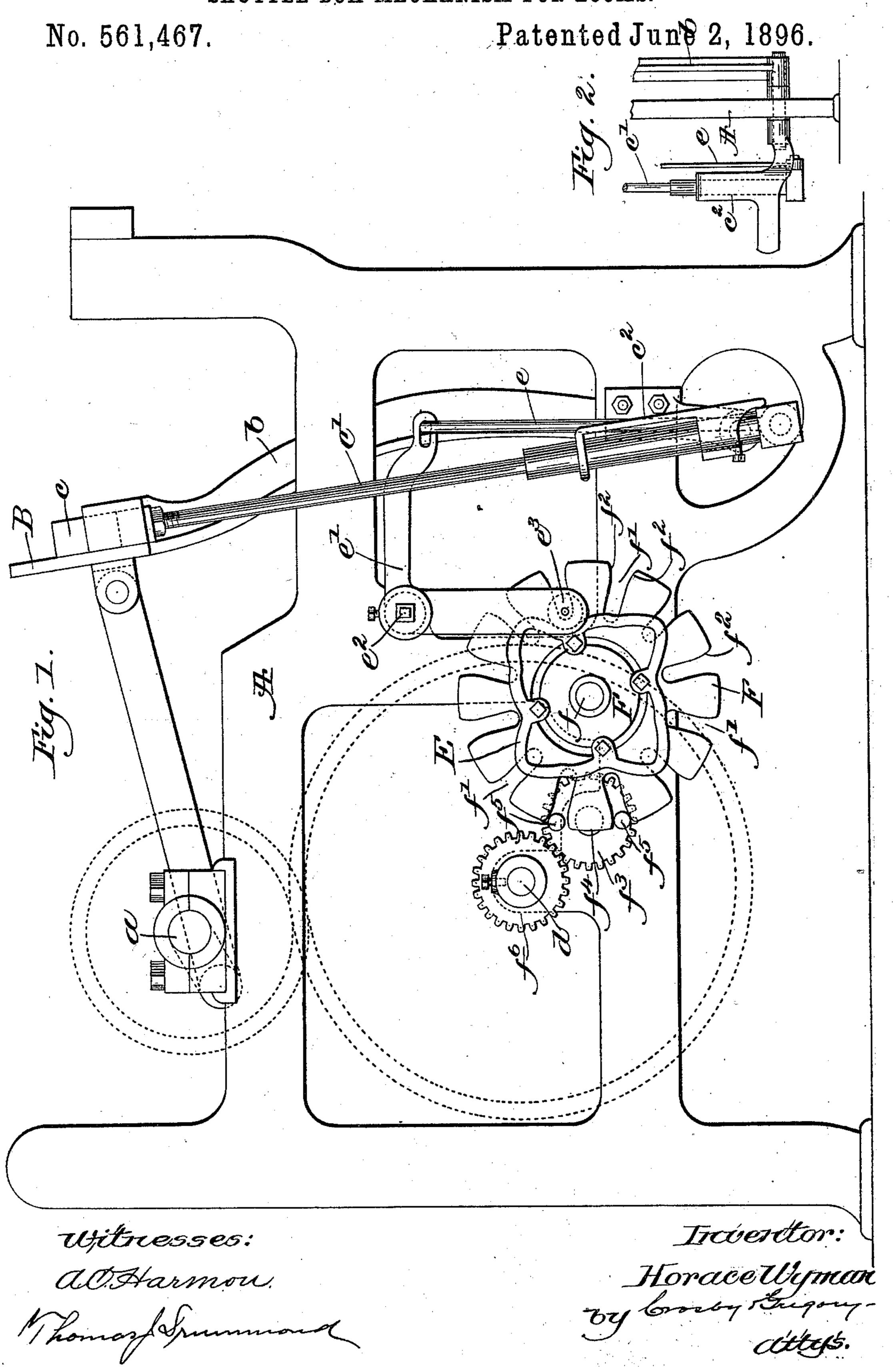
H. WYMAN.
SHUTTLE BOX MECHANISM FOR LOOMS.



## United States Patent Office.

HORACE WYMAN, OF WORCESTER, MASSACHUSETTS, ASSIGNOR TO THE CROMPTON LOOM WORKS, OF SAME PLACE.

## SHUTTLE-BOX MECHANISM FOR LOOMS.

SPECIFICATION forming part of Letters Patent No. 561,467, dated June 2, 1896.

Application filed January 11, 1896. Serial No. 575,095. (No model.)

To all whom it may concern:

Be it known that I, Horace Wyman, of Worcester, county of Worcester, State of Massachusetts, have invented an Improvement in 5 Shuttle-Box Mechanism for Looms, of which the following description, in connection with the accompanying drawings, is a specification, like letters on the drawings representing like parts.

This invention relates to mechanism for changing or shifting the shuttle-boxes of a 100m.

In the operation of a modern loom the time in which the shuttle-boxes must be shifted is 15 very short, and the box mechanism must act

quickly and with precision.

Frequently the boxes and their connections are of considerable weight, and the shafts, wheels, and other parts required to move the 20 same must necessarily be heavy and strong, so heavy, in fact, that difficulty is found in operating them at the speed and with the accuracy desired.

My present invention has for its object to 25 provide an improved and novel box mechanism wherein the number and speed of operation of the several parts are reduced to a

minimum.

In accordance with my invention the box 30 operating or shifting member is actuated by a cam, as heretofore, said cam, however, being in my invention rotated by a wheel provided with a plurality of slots or notches to enable it to be intermittingly rotated by a driv-35 ing-wheel provided with a plurality of projections adapted to enter said slots or notches.

In the preferred embodiment of my invention the slotted or notched wheel will be of considerable diameter and provided with a 40 large number of slots in order that each stepby-step or rotative movement of the wheel

may be as slight as possible.

By my invention a continuously-rotating wheel may be made to engage the notched 45 wheel for nearly one-half of each rotation of the former, yet act through only about onequarter of each rotation to actually move the said notched or slotted wheel and its cam, so that a quick intermitting cam movement may 5° be obtained from a continuously and relatively slow rotating driving-wheel.

In the drawings, Figure 1, in end elevation, shows a sufficient portion of a loom to enable my invention to be understood, Fig. 2 being a detail to be referred to.

In the particular embodiment of my invention illustrated in the drawings the frame, (indicated at A,) the crank-shaft a, connected with and to vibrate the lay B, the lay-swords b, the shuttle-box c, its lift-rod c', the swinging 60 bracket or frame  $c^2$ , connected with and vibrated with the lay and in which the lift-rod c' slides, and the main or driving shaft d, geared to and operating the crank-shaft, are and may be of suitable or usual construction 65 and operation, and need not be herein described in detail.

In the present instance of my invention the shuttle-box lift-rod c' is connected by a link e(see Fig. 2) with one arm of the bell-crank 70 lever or shifting member e', fulcrumed at  $e^2$ on the frame and having its other arm provided in the present instance with a suitable roller-stud  $e^{3}$ , held in operative contact with the cam E by the weight of the shuttle-box 75 on its lift-rod, or a suitable spring may be em-

ployed, if desired.

The cam E, as herein shown, is provided with a plurality (shown as four) of projections or tappets, of the same or, as herein shown, 80 of different heights, to impart the desired movement to the shuttle-box, ranging from its highest to its lowest position, and these tappets or projections are preferably so formed or shaped, as shown, as to reduce so 85 far as possible all unnecessary movement of the cam, and also of such shape as to most quickly shift the box when the cam is changed. For instance, the depression between two tappet projections is of a size nearly or quite fit- 90 ting the roller  $e^3$ , so that the initial movement of the cam will act to move the roller and partially lift the shuttle-box, and the highest portion of the projection is made as pointed as is practicable in order that no unnecessary 95 movement of the cam shall take place when the roller has reached the highest point on any projection.

The cam E, as herein shown, is bolted or otherwise secured to the face of a wheel F, 100 loosely mounted upon a stud f, or it may be made fast upon a shaft, if desired, so long as

it is free to be rotated. This wheel F, as herein shown, is provided with a large number of radial slots or notches f', the entrances to which are made preferably slightly flaring, 5 as at  $f^2$ , said wheel being driven by the driving-wheel  $f^3$ , fast on a shaft, or it may be loosely mounted upon a stud  $f^4$  on the frame and provided with a plurality, preferably two, projections or pins  $f^5$ , adapted to enter the 10 slots f' in the wheel F to drive the latter.

The driving-wheel  $f^3$  may be geared to and driven by a mating wheel  $f^6$ , fast on the main

or driving shaft d of the frame.

In the present instance of my invention the 15 shaft d rotates once for each two picks of the loom, and a corresponding rotation is imparted to the driving-wheel  $f^3$ , as the two wheels are of substantially the same diameter. At each rotation of the driving-wheel  $f^3$ 20 one of its pins  $f^5$  enters a notch or slot f' in the slotted wheel F and turns the latter a distance represented by the distance between two slots on the wheel.

In the drawings one of the pins is shown 25 as just entering a slot and the other pin as just leaving another slot, and starting from this position of the driving-wheel the uppermost pin  $f^5$  will rotate for nearly one-eighth of a rotation of the driving-wheel before it 30 will begin to move the slotted wheel F. It will then move the wheel through substantially one-quarter of its rotation and will then permit the said slotted wheel to stand at rest for another one-eighth of a rotation before its 35 pin leaves the slot in the wheel, and, therefore, since the slotted wheel is held against movement during an eighth of a rotation before a pin leaves a slot, and after a pin leaves a slot for another one-eighth of a rotation, 40 while the other pin is entering a new slot, the wheel is held against movement for one-quarter of a rotation of the driving-wheel  $f^3$ , is then moved for a distance represented by onequarter of the rotation of the said driving-45 wheel and again rested for another one-quarter of a rotation of said wheel, and so on. The slotted wheel F therefore is given an intermitting rotary movement by and from a continuously-rotating driving-wheel, the

wheel. The driving-shaft d and driving-wheel  $f^3$  ro-55 tate at a relatively slow speed, and the slotted wheel F, being of relatively large diameter with a large number of slots f', receives only a comparatively short movement for each shifting of the shuttle-box, so that the wear 60 and tear of the parts is reduced to a minimum, and the high speed heretofore necessary in operating the parts of the box mechanism is entirely overcome. Furthermore, the pins working in the slots start the slotted wheel 65 easily and without shock, thereby differing from pawl-and-ratchet or other mechanism

50 slotted wheel having two movements and two

periods of rest, all of substantially the same

duration, during each rotation of the driving-

for imparting a relative intermitting motion to the cam-wheel.

It will be noticed that in the mechanism shown embodying my invention the axis of 70 the driving-wheel  $f^3$  lies inside of the peripheral line of the driving-wheel F. Hence one of the pins  $f^5$  enters its slot at the top of the driving-wheel before the other pin leaves its slot at the bottom of said wheel, so that the 75 driving-wheel F is always under the direct control of at least one of the pins  $f^5$ , whereas if the axis of the said driving-wheel was outside the peripheral line of the driving-wheel one of the pins must necessarily leave its slot 80 before the other pin can fully enter its slot, thereby leaving the driving-wheel for at least two periods in each rotation of the drivingwheel without any means of holding it in position, except some means independent of the 85 pins be provided to retain it positively in position during the periods when it is not engaged by one or the other of the pins.

My invention is not limited to the particular shape and construction of parts herein 90 shown, for it is evident the same may be varied without departing from the spirit and

scope of my invention.

Having described my invention, what I claim, and desire to secure by Letters Patent, 95 1S--

1. In a box mechanism for looms, the combination with a box-shifting member and a rotatable cam to actuate the same, of a slotted wheel connected with and to actuate said 100 cam, and a driving-wheel provided with two diametrically opposite projections to enter the slots in and impart an intermitting rotation to said slotted wheel, one of said projections entering its slot before the other pro- 105 jection leaves its slot, whereby said projections serve to retain said slotted wheel against movement between successive intermitting movements, substantially as described.

2. In a box mechanism for looms, the com- 110 bination with a box-shifting member and a rotating cam, provided with tappet-like projections of different height to shift the said shifting member into different positions, of a slotted wheel connected with and to actuate 115 said cam, and a driving-wheel provided with a plurality of projections to enter the slots in and impart an intermitting rotation to said slotted wheel, one of said projections entering its slot before another projection leaves 120 its slot, whereby said projections serve to retain said slotted wheel against movement between successive intermitting movements, substantially as described.

3. In a box mechanism for looms, the com- 125 bination with a box-shifting member and a rotatable cam provided with a plurality of tappet-like projections, of a wheel connected with and to rotate said cam, and provided with a plurality of slots, and a driving-wheel 130 having a plurality of projections to enter the slots in and to impart an intermitting rota-

tion to said slotted wheel, said driving and slotted wheels being arranged to cause one driving projection to enter its slot before another leaves its slot, whereby said slotted wheel is always under the direct control of said driving-wheel, substantially as described.

4. In a box mechanism for looms, the combination with a box-shifting member and a rotatable cam to actuate the same, of a wheel connected with and to actuate said cam, and provided with a plurality of slots having flaring entrances thereto, and a driving-wheel provided with a plurality of projections to en-

ter the slots in and to impart an intermitting rotation to said slotted wheel, the axis of said 15 driving - wheel being inside the peripheral line of said slotted wheel, substantially as described.

In testimony whereof I have signed my name to this specification in the presence of 20 two subscribing witnesses.

HORACE WYMAN.

Witnesses:

FREDERICK L. EMERY, EMMA J. BENNETT.