

(No Model.)

3 Sheets—Sheet 1.

E. TILSTON.

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TO PREPARING MACHINES.

No. 551,800.

Patented Dec. 24, 1895.

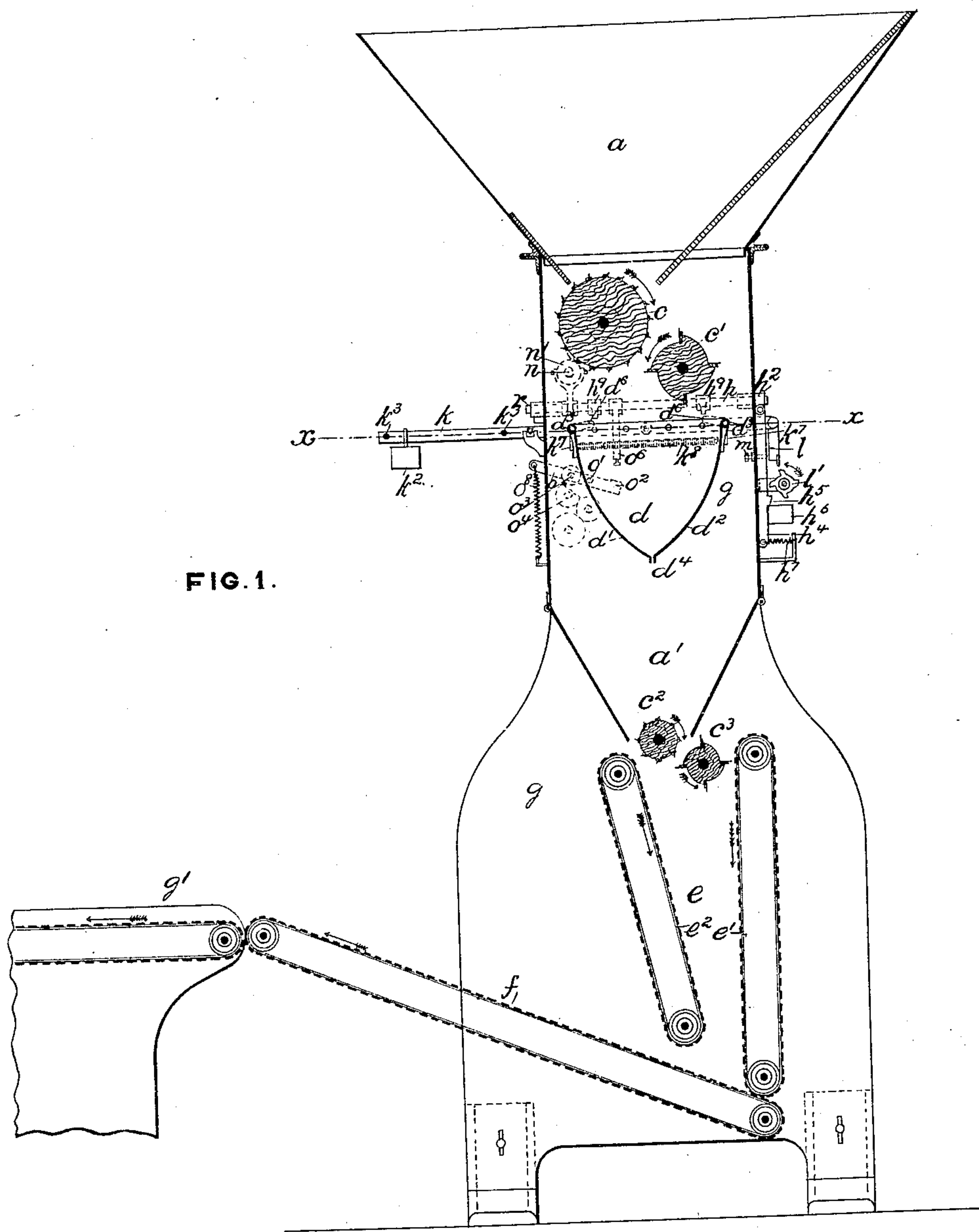


FIG. 1.

Witnesses.  
Desider Franka  
Chas. A. Muzzy.

Inventor  
Edward Tilston  
by Wm. H. Babcock  
Attorney

(No Model.)

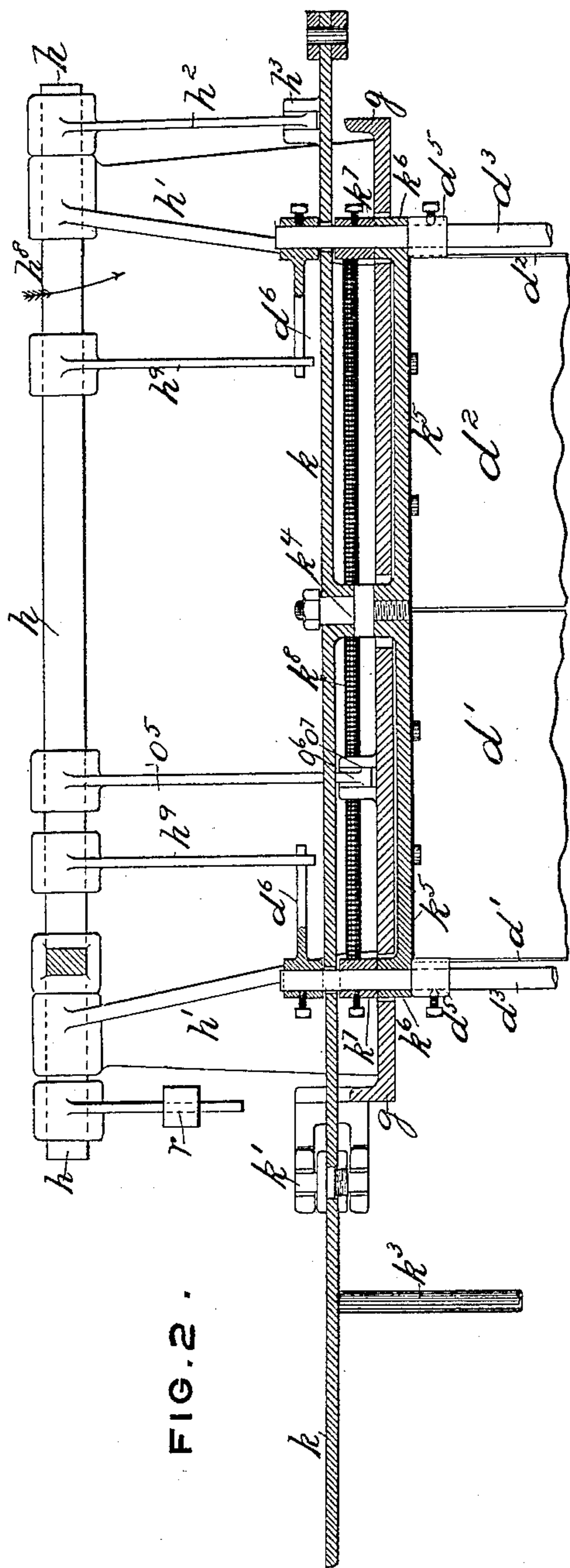
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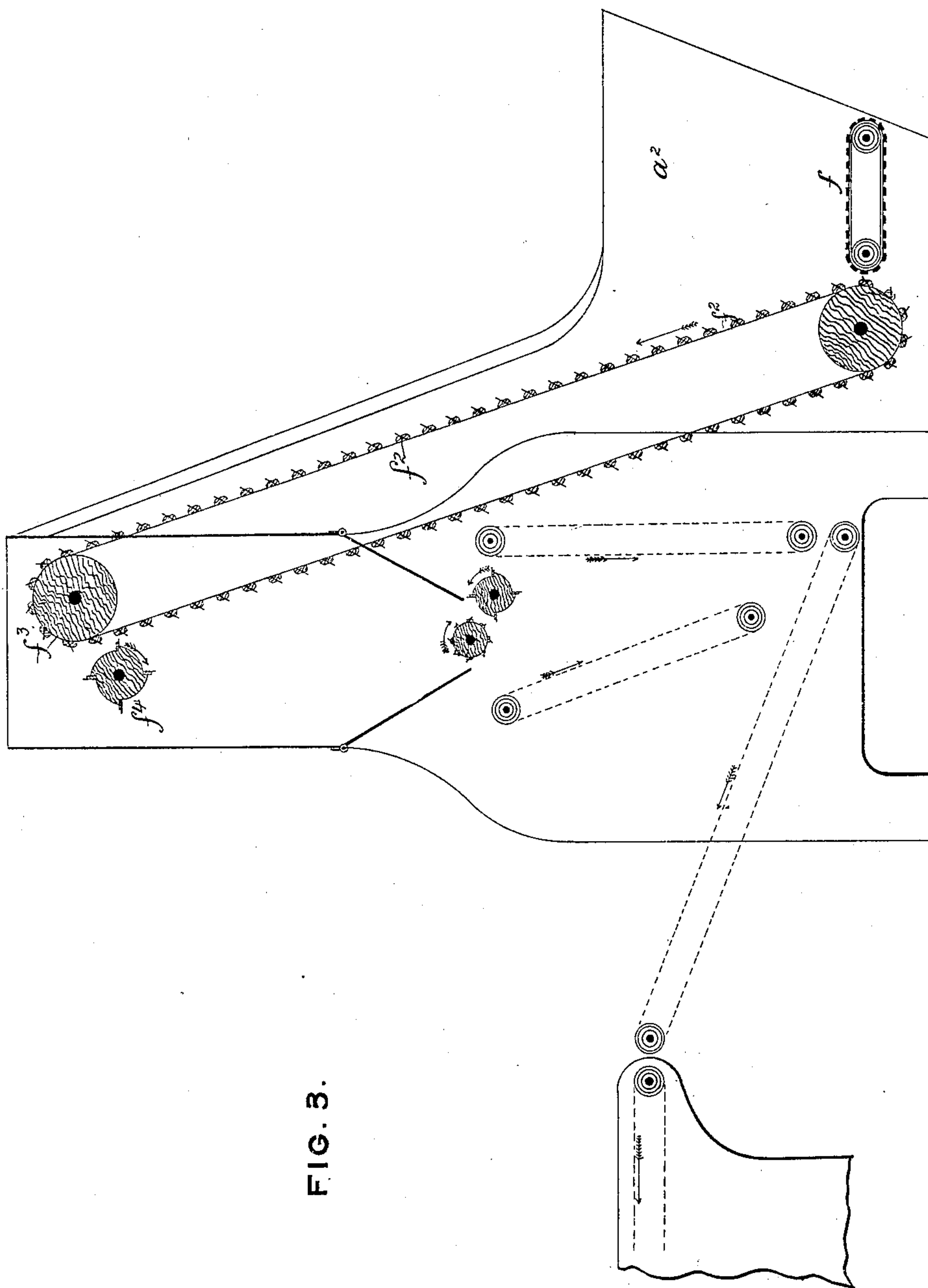


FIG. 3.

Witnesses.  
Desider Lankka  
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Edward Tilston  
by Wm H Babcock  
Attorney



# UNITED STATES PATENT OFFICE.

EDWARD TILSTON, OF MANCHESTER, ENGLAND.

APPARATUS FOR AUTOMATIC SUPPLY OF FIBROUS MATERIAL TO PREPARING-MACHINES.

SPECIFICATION forming part of Letters Patent No. 551,800, dated December 24, 1895.

Application filed August 26, 1895. Serial No. 560,562. (No model.)

*To all whom it may concern:*

Be it known that I, EDWARD TILSTON, a subject of the Queen of Great Britain and Ireland, and a resident of 2 Bates Street, Longsight, Manchester, county of Lancaster, England, have invented certain Improved Apparatus for the Automatic Supply of Fibrous Material to Preparing-Machines, of which the following is a specification.

My said invention relates to improvements in machinery for the automatic supply of definite quantities of raw or partly-prepared fibrous materials, such as cotton and wool, to the lattice or feed-table of machines used in opening, beating, cleaning, or otherwise preparing such materials.

For more clearly setting forth my said invention I will describe the same with reference to the accompanying drawings, in which—

Figure 1 represents a vertical central section of a machine embodying my invention. Fig. 2 represents a plan view, partly in section, of the mechanism for operating the scale-pan; and Fig. 3 represents a vertical section of a modified combination and arrangement of the hopper and its feeding devices.

The hopper for the fibrous material to be acted on is designated *a*.

The material is drawn from the hopper *a* by the roller *c* and clearer *c'*, whence it falls into the closed balance or scale-pan *d*. The latter is made in two halves *d'* *d''*, each mounted on a rocking shaft *d<sup>3</sup>*. When a certain predetermined quantity of material has accumulated in the scale-pan *d* the shafts *d<sup>3</sup>* make a partial rotation about their axes (by the means hereinafter described) in such a manner as to cause the two halves *d'* and *d''* of the scale-pan to diverge, whereupon the material falls through at *d<sup>4</sup>* into the lower hopper or chute *a'*, whence it is drawn by the spiked roller *c<sup>2</sup>* and stripper *c<sup>3</sup>* and delivered to the feed-box *e*. The sides *e'* *e<sup>2</sup>* of the feed-box are in some cases inclined toward each other in such a way as to make the lower part of the feed-box narrower than the upper part. It frequently happens that the material varies much in bulk for a given weight, while it is desirable to deliver it to the feed-lattice of as uniform density as possible.

By having a feed-box with inclined sides the combined effect of gravity and the inclined

planes thus presented will tend to force the heterogeneous layers together, compressing all and increasing the approach to uniformity in density. To facilitate the descent and discharge of the material one or both sides of the box may be provided, as shown in Fig. 1, with moving devices—for example, endless traveling sheets or lattices. As the said material leaves the box *e* it falls on a creeper or conveying-lattice *f*, which carries it to the feed-table *g'* of the opening or other machine to which the material has to be supplied.

It will usually be most convenient to arrange the hopper *a* under an opening in an upper floor through which the material is discharged. Where this is not convenient I adopt the arrangement shown in Fig. 3. The hopper is placed at *a<sup>2</sup>* and the material therein is led by a creeper *f* to a spiked traveling sheet or lattice *f<sup>2</sup>*, which carries the material upward over the roller *f<sup>3</sup>*, where the material is stripped by a stripper *f<sup>4</sup>*. The rest of the machine is as herein described with reference to Fig. 1.

The mechanism for operating the scale-pan is shown in elevation in Fig. 1 and in plan to a larger scale in Fig. 2. In these figures *g* is one of the main side frames of the machine between which the hopper, the rollers, and the feed-box lie. There is one such side frame at each side, to one of which the mechanism for operating the scale-pan is attached.

Fig. 2 is a section on the line *x x* of Fig. 1; but for facility of description the shaft *h* and the levers thereon are also shown, although they are above the level of the plane represented by *x x*.

The scale-pan lever or beam *k* has its fixed fulcrum at *k'* on a bracket secured to the side of the machine. A sliding balance-weight *k<sup>2</sup>* is provided, which not only balances the weight of the lever *k*, the scale-pan and all the mechanism connected therewith, but besides furnishes an excess weight corresponding to the weight of material which it is desired to accumulate in the scale-pan *d* before the latter opens.

There is a balance-beam *k* at each side of the machine connected to each other by cross-bars or stretchers *k<sup>3</sup>* to insure their parallel action.

The cross-shafts *d<sup>3</sup>* are carried by the beams



5  $k$ , in which they can freely rotate. A bolt  $k^4$   
 pivoted in the beam  $k$  carries a bar  $k^5$ , through  
 the ends  $k^6$  of which the shafts  $d^3$  can freely  
 rotate. A sheet curtain  $k^7$  is secured by  
 screws to the bar  $k^5$  and prevents the material  
 from falling out at the sides of the scale-pan  $d$ .  
 The scale-pans  $d'$   $d^2$  are secured to brackets  
 10  $d^5$  mounted on the shafts  $d^3$  and secured  
 thereto by set-screws, so as to oscillate there-  
 with. Suitable slots are provided in the  
 frame sides  $g$  to admit the bosses of the bar  
 $k^5$  and to allow them to freely rise and fall  
 therein as the beam  $k$  oscillates about its ful-  
 crum.

15 Levers  $k^7$  are secured by set-screws to the  
 shafts  $d^3$  and have their ends connected by a  
 spring  $k^8$ , so as to keep the scale-pan closed,  
 unless some force is applied to open them  
 positively against the action of the spring  $k^8$ .

20 Hinged at one end of the beam  $k$  is a swing-  
 bar  $l$ , which when the scale-pan is closed is  
 held up so that its lower end is just out of  
 reach of a revolving spider-shaft  $l'$ .

25 At one side only of the machine is a shaft  
 $h$  free to rotate in bearings  $h'$  secured to the  
 side frame  $g$ .

Secured to one end of the shaft  $h$  is a lever  
 $h^2$ , having a universal joint  $h^3$ , from which  
 depends a strut  $h^4$ , having a projection  $h^5$   
 30 resting on a block  $h^6$ . A spring  $h^7$  tends to  
 keep the projection  $h^6$  in position. The weight  
 of the levers and other parts connected to the  
 shaft  $h$  tends to cause the shaft to turn in the  
 direction of the arrow  $h^8$ . (Shown in Fig. 2.)

35 A spring or weighted lever  $r$  may, however,  
 be applied to augment this tendency and pre-  
 vent sticking.

When the weight of the material in the pan  
 $d$  has reached such a point as to overcome  
 40 the weight  $k^2$ , the latter will rise and the other  
 end of the beam  $k$  will fall, bringing the lower  
 end of the hang-bar  $l$  into contact with the  
 rotating spider-shaft  $l'$ . The bar  $l$  will then  
 be driven in toward the machine, and, meet-

45 ing a bolt  $m$  on the strut  $h^4$ , the projection  $h^5$   
 will slip off the block  $h^6$ , whereupon the strut  
 $h^4$  will no longer prevent the turning of the  
 shaft  $h$  in the direction of the arrow  $h^8$ . On  
 the turning of the shaft  $h$  in the said direc-  
 50 tion the arms  $h^9$  of the shaft  $h$  will strike the  
 arms  $d^6$  and depress them, thereby partially  
 rotating the shafts  $d^3$  and causing the scale-  
 pans  $d'$   $d^2$  to diverge and to discharge their  
 contents into the hopper  $a'$ . The turning of

55 the shaft  $h$  also stops the rotation of the  
 spiked roller  $c$ , (or the roll  $f^3$  of Fig. 3.) The  
 roller  $c$  is driven by toothed gearing from a  
 rotating shaft  $n$ , on which is a clutch  $n'$ , con-  
 trolled by a lever from the shaft  $h$  in such a  
 60 way that as the shaft turns to open the scale-  
 pan the clutch is disengaged and as the shaft  
 is put back the clutch engages again and the  
 spiked roller (or the roller  $f^3$ ) again rotates.

65 After having discharged its contents the pan  
 $d$  remains open a certain predetermined time,  
 when it is automatically closed by means  
 about to be described.

A lever  $o$ , having a fixed fulcrum at  $o'$  on  
 the same side of the machine as the shaft  $h$ ,  
 is provided with a spring  $o^8$ , which continu- 70  
 ally tends to pull down the outer end of the  
 lever  $o$  and to elevate the inner end  $o^2$ . On  
 the lever  $o$  is a roller  $o^3$ , taking onto a slowly-  
 rotating cam  $o^4$ . On the shaft  $h$  is a lever  $o^5$   
 connected to a plunger  $o^6$  working in guides 75  
 $o^7$ . So long as the strut  $h^4$  is in engagement  
 with the block  $h^6$  and the lever  $o^5$  is up the  
 plunger  $o^6$  is kept clear of the end of the le-  
 ver  $o^2$ . The swell of the cam  $o^4$ , pressing on  
 the roller  $o^3$ , keeps the end of the lever  $o^2$  de- 80  
 pressed; but when the depression in the cam  
 comes under the roller  $o^3$  the spring  $o^8$  elevates  
 the point of the lever  $o^2$ . By the opening of  
 the scale-pan  $d$  through the rocking of the  
 shaft  $h$  the plunger  $o^6$  is depressed, and the 85  
 period of rotation of the cam  $o^4$  is so arranged  
 that at any given interval of time after the  
 opening of the scale-pan  $d$  the depression of  
 the cam shall come under the roller  $o^3$ , when  
 the point  $o^2$  of the lever instantly rises and 90  
 lifts the plunger  $o^6$ , thereby rocking the shaft  
 $h$  backward. By this backward rocking the  
 scale-pan is closed, the strut  $h^4$  is lifted into  
 its first position, the clutch  $n'$  is brought into  
 engagement, all parts revert to their original 95  
 positions, and the supply of material recom-  
 mences.

By varying the toothed change-wheels  
 which drive the cam  $o^4$  the rotation of the  
 latter and thereby the interval of time elaps- 100  
 ing between the successive closings of the pan  
 $d$  can be regulated at will to suit the particu-  
 lar rate of feed suitable for the machine to be  
 supplied or the material operated upon.

The arrangement of gearing for driving the 105  
 various rollers from the main driving-shaft,  
 (other than that described,) the feed-lattices,  
 and the construction of these elements are not  
 shown, as they may be arranged in various  
 well-known ways and form no part of my said 110  
 invention.

What I claim as my invention, and desire  
 to secure by Letters Patent, is—

1. In a machine for automatically supply-  
 ing weighed quantities of fibrous materials 115  
 the combination of the diverging scale pans  
 $d'$   $d^2$  the hopper  $a'$  the continuously rotating  
 roller and stripper  $c^2$   $c^3$  and the feed box  $e$  hav-  
 ing sides inclined to each other substantially  
 as set forth and shown. 120

2. In a machine for automatically supply-  
 ing fibrous materials the combination of an  
 intermittently acting weighing apparatus  
 with a feed box having sides inclined down-  
 ward toward each other and provided with 125  
 traveling sheets for taking forward and de-  
 livering the material continuously substan-  
 tially as set forth.

3. The scale-pan consisting of two inde-  
 pendently pivoted sections held together by 130  
 a spring, in combination with means of sup-  
 plying the same with material and devices for  
 opening the said scale-pan at intervals to dis-  
 charge the material therefrom, the said de-



vices consisting of a lever  $k$ , a weight  $k^2$  balancing the said pan and lever as well as the connected mechanism and a predetermined amount of material to be held in the said pan,  
5 the hang bar  $l$ , the spider-shaft  $l'$  arranged to be struck by the said hang bar as the scale pan descends, a rock-shaft  $h$  provided with arms  $h^9$  and also with a stop which is withdrawn by the action of the said hang bar, and  
10 shafts on which the sections of the scale-pan are respectively pivoted, these shafts being

arranged to turn so as to open the said scale-pan and provided with arms  $d^6$  arranged to be struck by the said arms  $h^9$  substantially as set forth. 15

In testimony whereof I have signed my name to this specification in the presence of two subscribing witnesses.

EDWARD TILSTON.

Witnesses:

JOHN HALL,  
JOSEPH BENTON.