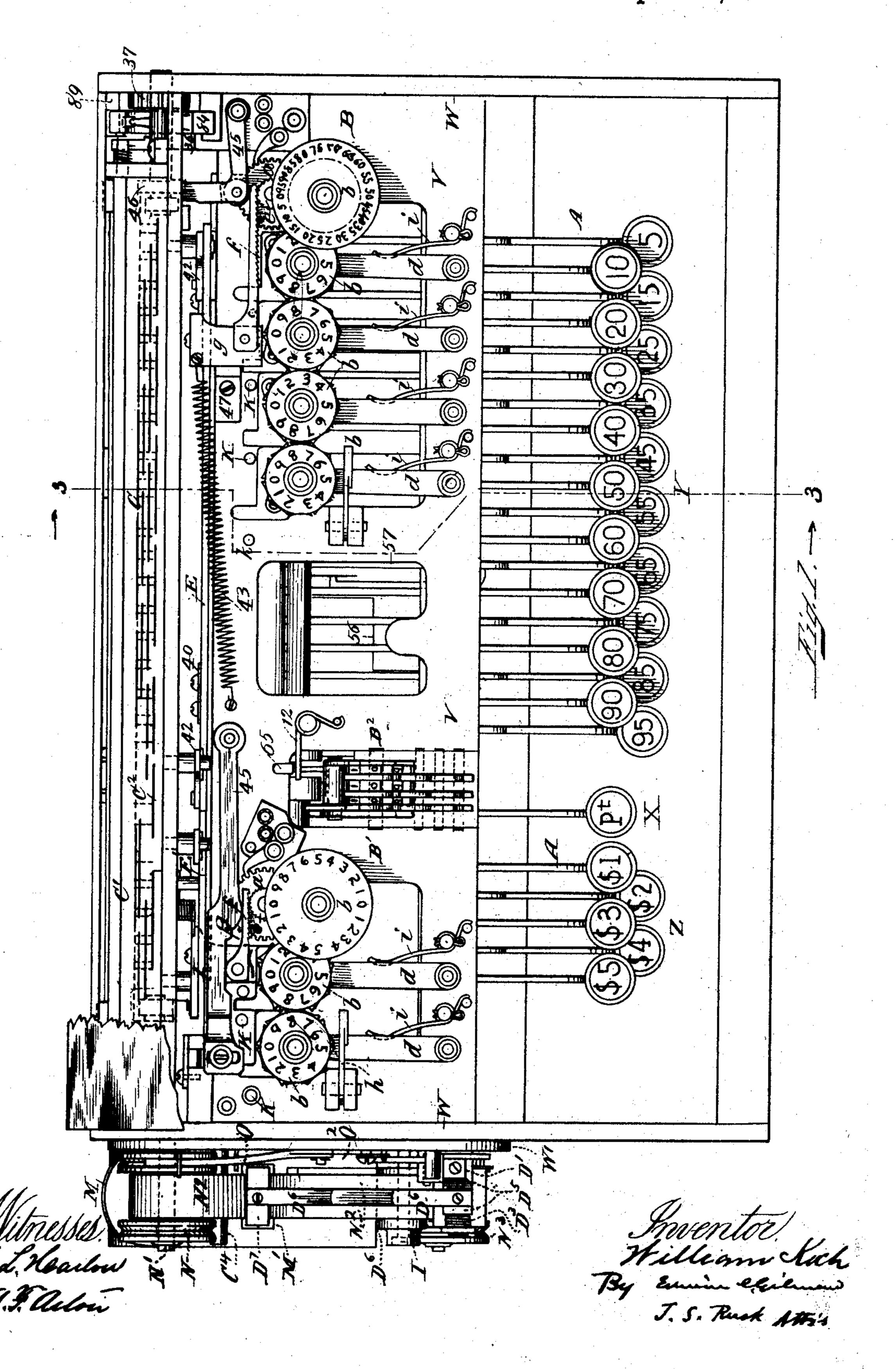
### W. KOCH. CASH REGISTER AND RECORDER.

No. 505,555.

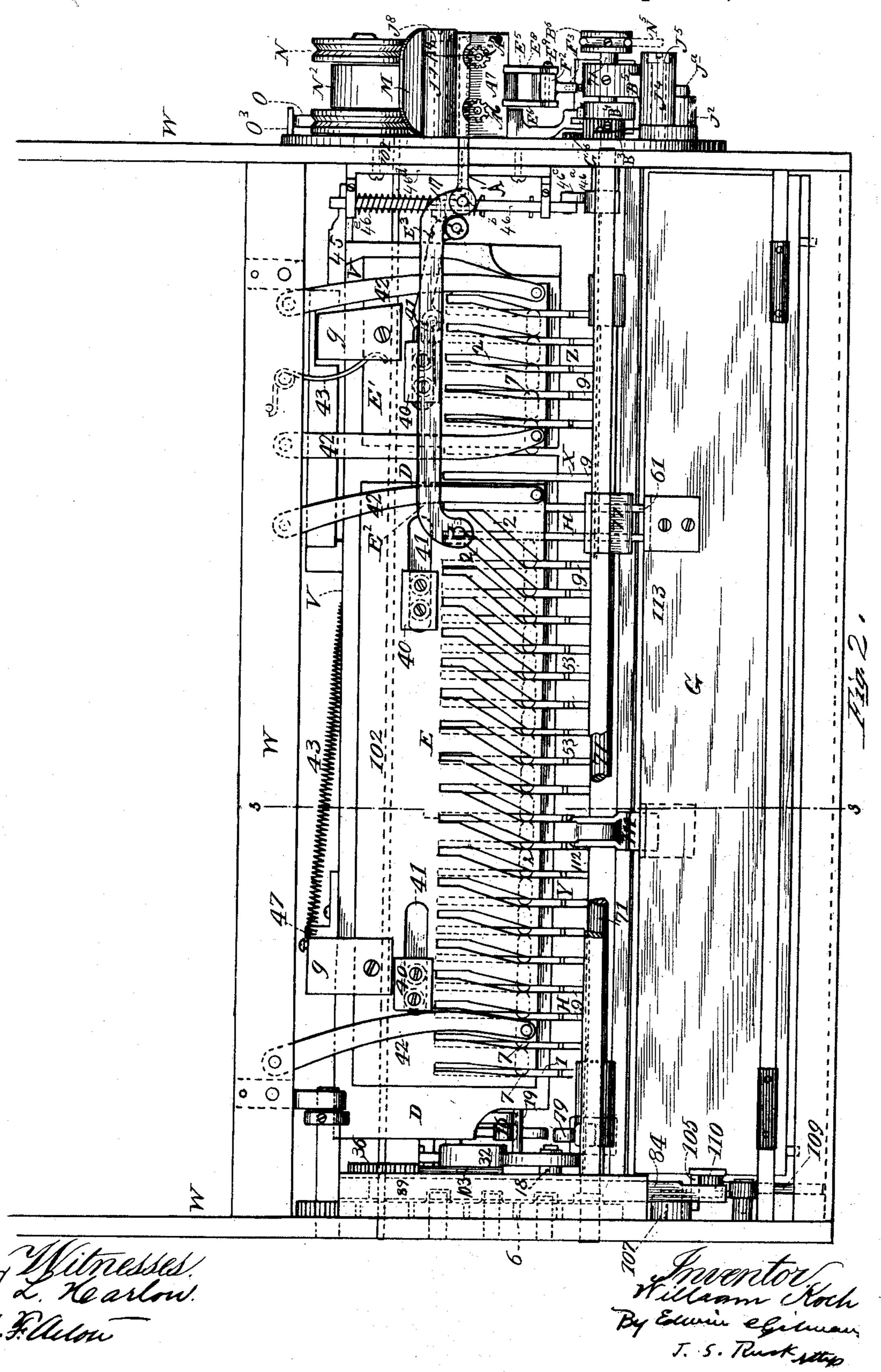
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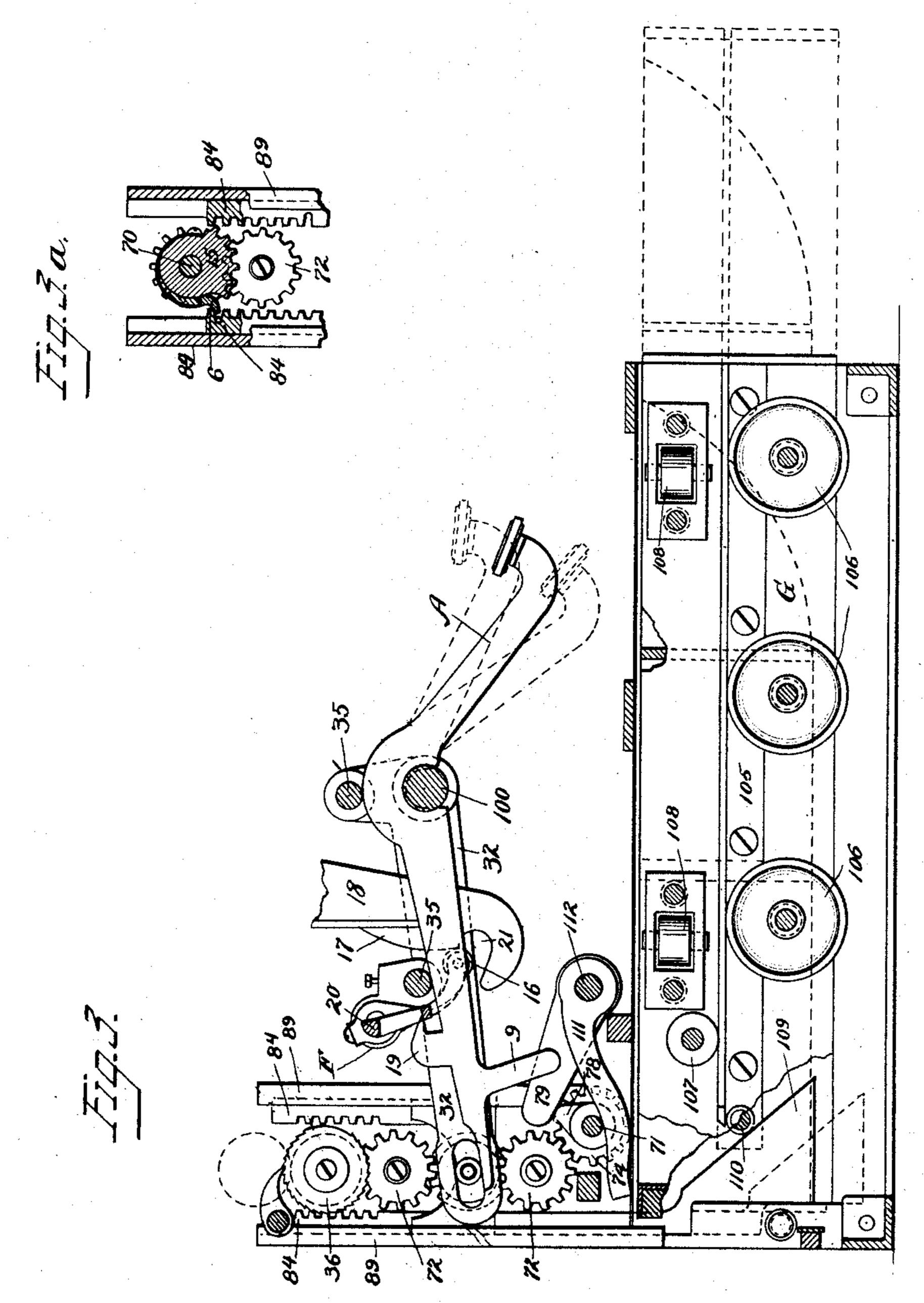
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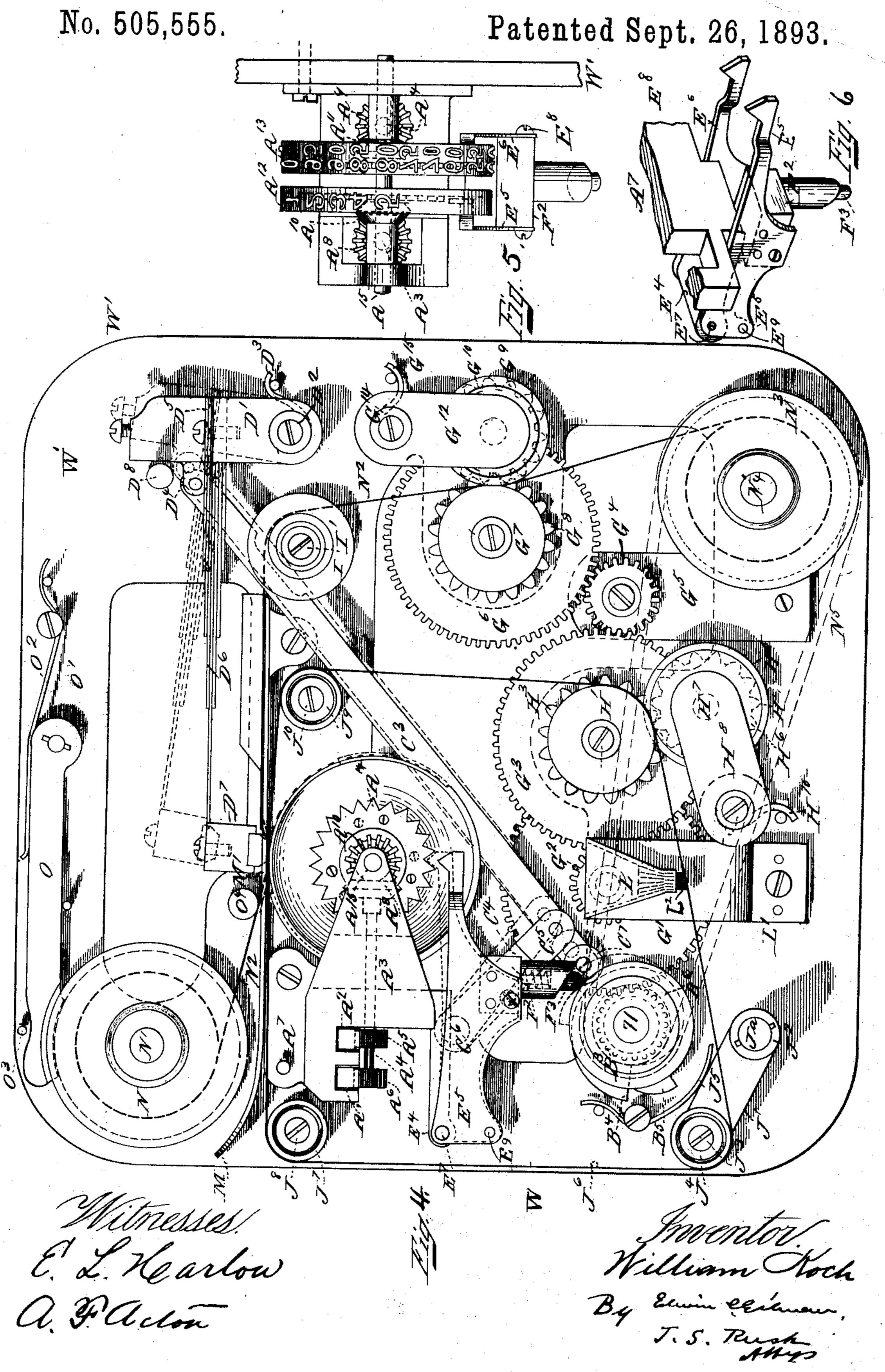
Patented Sept. 26, 1893.



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CASH REGISTER AND RECORDER.



#### United States Patent Office.

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#### CASH REGISTER AND RECORDER.

SPECIFICATION forming part of Letters Patent No. 505,555, dated September 26, 1893.

Application filed February 29, 1892. Serial No. 423, 268. (No model.)

To all whom it may concern:

Be it known that I, WILLIAM KOCH, a citizen of the United States, residing in the city, county, and State of New York, have invent-5 ed a certain new and useful Improvement in Cash-Registers, of which the following is a description, reference being had to the accompanying drawings, forming part of this speci-

fication. My invention relates more particularly to machines of the character shown and described in my pending application, Serial No. - 399,640, filed July 15, 1891, and consists chiefly in the application to that machine of a print-15 ing mechanism by which the amounts indicated and registered by the operations of the various keys may be also printed upon a paper strip carried by the machine. In the machine shown in my prior application the values 20 of the operated keys in each set were added into a common total, so that at the end of a day's business the proprietor could ascertain the gross amount of the sales by inspecting the register, but he could not ascertain the 25 character or amounts of the different sales. It is desirable in this class of machines, for reasons familiar to those experienced in the sale and use of such machines, that the proprietor shall be able at the end of a day's 30 business to readily ascertain not only the total amount of the sales but to have before him a record of the individual sales, so that he may know just how many sales of differ-

ent amounts have been made. To that end my present invention consists in equipping my former machine with a printing mechanism by which a record of such individual sales may be automatically made and preserved. In combining such a mechanism 40 with the machine of my prior application, however, I have produced substantially a new machine, containing combinations of parts and modes of operation which are broadly new, and my present invention is therefore 45 not restricted, in its broader scope, to the particular construction of the machine shown in my prior application, or the particular construction of the printing mechanism which has been combined with it, but contemplates 50 broadly the new combinations and modes of

without further limitation than is therein expressed.

Having thus premised the general nature of my invention, I will proceed to describe, first 55 the general construction of the machine as shown and described in my prior application. and next the combination of the printing mechanism therewith, reference being had to the accompanying drawings, in which-

Figure 1 represents a top plan view of the machine, removed from its casing, and with some of the parts broken away to expose others beneath; Fig. 2 a rear elevation of the machine, with the indicating mechanism re- 65 moved; Fig. 3 a vertical section of the machine approximately on the line 3-3 of Fig. 1 with one of the operating keys partly depressed; Fig. 3a a detail of the segment forming part of the mechanical movement for converting 70 the reciprocating movements of the keys into rotary movement in a shaft; Fig. 4 an enlarged side elevation of the printing attachment upon the left hand side of the machine; Fig. 5 a detail front elevation of the two type 75 wheels and associated parts; and Fig. 6 a detail perspective view of the locking dogs for the type wheels and associated parts.

The same letters and numerals of reference are used to indicate identical parts in all the 80

figures.

Inasmuch as the general construction of the machine is described in detail in my aforesaid pending application I will here give only such brief description of the parts and their 85 mode of operation as may be necessary to an understanding of my present invention. The operating levers consist of bent levers A fulcrumed on a horizontal rod 100, Figs. 1 and 3. They fit in grooves in said rod, or are prop- 90 erly spaced upon it by interposed collars, and their rear ends are confined and adapted to have their vertical play in guide slots in a fixed plate D of the framework, Fig. 2. Hung upon the rod 100 is a swinging frame com- 95 posed of two side arms 32 rigidly connected in rear of the fulcrum rod 100 by a rod 35 overlying the keys, and having their vertically extended front ends also rigidly connected by a second rod 35 immediately over 100 the fulcrum rod 100. This latter rod serves operation set forth in my respective claims I to hold the key levers A in position in the

grooves in the rod 100, the upper edges of the keys against which the rod 35 bears being curved in the arc of a circle struck from the center of the rod 100. The two side arms 32 5 and the cross rods 35 constitute a rigid frame hung upon the rod 100. In the normal position of the parts the rear rod 35 rests a slight distance above the upper edges of the keys, and whenever any key is operated it contacts ro with the rod after it has been moved slightly from normal position, and lifts the swinging frame with it. The keys are provided upon their upper sides in rear of the rod 35 with hooks 19 adapted to co-operate with a coup-15 ling bar 20 loosely hung between ears F upon the upper edges of the side arms 32, said coupling bar having at one end a downwardly and forwardly curved arm carrying a roller 16 adapted to co-operate with a cam 17 upon 20 a fixed plate 18 depending from an upper part of the framework, and with a recess 21 in said plate below the cam 17, to cause an operated key to become hooked to the bar 20, and thereby coupled to the swinging frame, 25 as soon as it has been substantially displaced from normal position, as will be readily understood, and as is explained at length in my aforesaid pending application. The key levers A are in this instance divided into two 30 sets, Y and Z, the former representing cents in multiples of five from 5 to 95, and the latter representing dollars in multiples of one from 1 to 5, with an intermediate special key X adapted to operate an independent regis-35 tering mechanism B2. The front ends of the key levers in the two sets are preferably so bent as that the finger buttons of the alternate keys will stand in two horizontal lines or banks, as shown, and as is usual in this Upon the rear side of the fixed guide plate D heretofore referred to, Fig. 2, are mounted two longitudinally movable plates E and E', the former overlying the rear ends of the lying the rear ends of the keys in the dollar bank Z. The plate E is provided with two

40 class of machines. 45 keys in the cents bank Y, and the latter overlongitudinal slots 41 through which pass screw studs upon the rear side of the fixed 50 plate D, said studs being surrounded with anti-friction collars, and having plates 40 secured upon their outer ends, which plates serve to hold the plate E in proper position relatively to the plate D. In addition to its 55 support upon the studs above referred to the plate E is also hung to the lower ends of a pair of links 42 pivoted at their upper ends to the cross bar W of the framework. The plate E' is simi'arly supported upon the plate 60 D by studs and by a second pair of links 42, as seen in Fig. 2. The plate E is provided with a series of differentially inclined slots 2, one for each key in the cents bank Y. The lower ends of these slots, when the plate is 65 in its normal position, rest immediately over and in line with the respective keys, so that |

its rear end in rising will enter its co-operating slot. The varying inclination of these slots is such that when the rear end of any 70 key lever is moved upward through the entire length of its co-operating slot it will move the plate E longitudinally a distance proportionate to the value of such key. Thus, the slot at the extreme left hand end 75 of the plate E in Fig. 2 is but slightly inclined from vertical position. This slot cooperates with the extreme right hand key in the cents bank Y, Fig. 1, so that the operation of said key will move the plate E but a 80 slight distance. Taking this distance as a unit of movement, the next slot in the plate E is inclined from vertical position far enough to cause its co-operating key to move the plate E two units of distance, and so on 85 in regular order to the nineteenth slot at the extreme right hand end of the plate E in Fig. 2, which slot is so inclined as to cause the operation of its co-operating key to move the plate E nineteen units of distance. The plate 90 E has secured to it an angle plate g whose upperend projects forward in horizontal position over the main top plate V of the framework, Fig. 1. The plate g has pivoted to it a ratch-bar f which co-operates with a ratchet e fast upon 95 the master wheel a of a train of registering. wheels B, and to which the differential movements of the plate E under the operations of different keys are transmitted, as will be readily understood and as is explained in de- 100 tail in my aforesaid application. A coiled spring 43 connected at one end to the plate g and at the other to the fixed frame plate Vyieldingly holds the plate E in and returns it to normal position. The plate E' is pro- 105 vided with a similar set of differentially inclined slots 2, one for each key in the dollar bank Z, and the movements of said plate are transmitted to a second registering mechanism B' through the medium of a plate g, ratch 110 bar f, ratchet wheel e and master wheel a, as in the case of the first mentioned registering mechanism.

The inclined slots in the plates E and E' terminate in vertical portions at both their 115 upper and lower ends, so that the rear end of an operated key will enter its co-operating slot and become fully engaged with the plate E before it begins to move the latter, and after it has passed through the inclined por- 120 tion of its slot and given the plate E its full movement in one direction the key will have a further idle movement in the vertical upper end of its slot, and the plate E will remain stationary. So, when the key is released and 125 its rear end begins to descend it will be moved downward some distance before the plate E will begin to return to normal position, and the plate will be completely returned to such position and come to rest before the key is 130 disengaged from the vertical lower end of its slot. This provision for idle movement of the operated key at the end of its positive when the front end of any key is depressed I stroke, after it has given the differential plate

its full movement corresponding to the value of the key, is especially important in connection with my present invention, since these plates are employed to set the type wheels 5 during the positive strokes of the keys, and the latter end of such stroke of the operated key, after the plate has come to rest, is employed to actuate the platen to print the number which the movement of the plate has brought to the printing point; all as will be hereinafter more particularly referred to.

As seen in Fig. 3 the rear ends of the key levers are reduced in size and shaped to fit the slots in the plates E and E', and the ad-15 justment of the parts is such that there is no room for lateral play between the sides of the key and the walls of its slot, and independent movement of the plates, and consequent inaccurate operations, is prevented. The inclined portions of the slots are wider than the vertical portions, the latter having simply to accommodate the thickness of the key while the former must accommodate more or less of the width of the key, according to

25 the inclination of the slot. The printing mechanism and the means for transmitting the differential movements of the plates E and E' to the type wheels or carriers, and the means for actuating the platen 30 and co-operating parts, may now be described. The printing mechanism is mainly supported upon a plate W', Figs. 1, 2, 4 and 5, secured to the left hand side frame of the machine. As seen in Figs. 4 and 5 the plate W'has se-35 cured to it a bracket plate A7, in the pointed forward end of which is mounted one end of a rod or shaft A<sup>15</sup> whose opposite end is secured in the plate W', Fig. 5. Loosely mounted upon this rod A<sup>15</sup> are the two type wheels 40 A<sup>12</sup> and A<sup>13</sup>, the former bearing upon its periphery five type numbers representing multiples of one from 1 to 5, and the latter bearing a series of type numbers in multiples of five from 5 to 95 inclusive, and in this instance also 45 a zero type. Each type wheel consists of a thin circular disk, Fig. 4, and a band or flange secured to or formed integral with it and upon whose surface are formed the type numbers. Secured to the hub of the wheel A12 is a bev-50 eled gear A<sup>10</sup> which meshes with a beveled gear A<sup>8</sup> fast upon a rod A<sup>8</sup> journaled near its rear end in the cross piece of the bracket A7 and having its extreme forward end journaled in a fixed collar surrounding the rod  $A^{15}$  between 55 the gear A<sup>10</sup> and the forward extension of bracket A7. The type wheel A13 has fast to its hub a similar beveled gear A11 which meshes with a beveled gear A9 fast upon a shaft A4, Fig. 5, said shaft A4 being journaled in the 60 bracket A7 and in a second fixed collar between the beveled gear A<sup>11</sup> and the right hand flange of the plate A7 by which the latter is secured to the plate W', Fig. 5. The shaft A4 has fast upon its rear end a pinion A6, Figs. 2 and 4 65 with which meshes a rack A', Fig. 4, whose left hand end is pivoted to the right hand end of an arm or link 17, Fig. 2, whose left 1

hand end is pivoted to the differential plate E, and the adjustment of the parts is such that whenever any key is operated the move- 70 ment given the plate E will be transmitted to the type wheel  $\bar{A}^{13}$  and the latter turned to bring to the printing point the number corresponding to the value of such key. The shaft A³ has fast upon its rear end a pinion 75 A<sup>5</sup>, Figs. 2 and 4 which meshes with a rack A2, Fig. 4 whose left hand end is fitted to the right hand end of a similar link 17, Fig. 2, which link is pivoted at its left hand end to the differential plate E', so that upon the oper-80 ation of any key in the dollar bank the movement imparted to the plate E' will be transmitted to the type wheel  $A^{12}$  and the latter turned to bring to the printing point the number corresponding to the value of such 85

key. The platen which co-operates with the type wheels, the means for advancing the paper strip, and the means for moving the inking ribbon, are all actuated by a rotary shaft 90 71', Figs. 2, 3, and 4, which extends entirely across the rear side of the machine, is journaled at its left hand end in the left hand side frame of the machine, or a bearing thereon, and projects at its right hand end 95 through the right hand side frame and plate W' which supports the printing mechanism. This shaft is given a complete revolution at each full operation of any one of the key levers, by means which may be briefly described 100 as follows, reference being had to Figs. 2, 3 and 3a: At its extreme left hand end, Fig. 2, just within the side frame of the machine, the shaft has fast upon it a gear 74, which, by a series of intermediate gears 72, is geared to 105 a rotary stub shaft 70 suitably journaled and supported at one end in the left hand side frame of the machine. This shaft has fast upon it, beside its gear 72, a toothed segment 15, Fig. 3a, adapted to co-operate with two 110 racks 84 formed upon the vertical arms of a vertically reciprocating frame arranged in guide ways 89 upon the side frame of the machine. The rack frame is provided with a laterally projecting stud 87 which fits in a 115 slot in the rear end of one of the side arms 32 of the swinging frame heretofore described, by means of which when the front end of any key lever is depressed the rack frame is lifted, and when said key lever is 120 returned to normal position the rack frame is carried down with its rear end. The segment 15 is provided with a recess in which is pivoted a spring-tooth 6 also adapted to co-operate with the racks. With the parts 125 in the position shown in Fig. 3a, when the front end of any key lever is fully depressed

and the rack bars 84 lifted the rear rack will

turn the segment 15 and shaft 70 forward a

tooth 6 will engage the forward rack 84, so

that upon the return of the key lever to nor-

mal position and downward movement of the

racks the forward rack will turn the segment

half revolution, at the end of which the spring- 130

and shaft 70 an additional half revolution, as will be readily understood, and as has been fully explained in my aforesaid prior application. In this manner the shaft 70 is given 5 a complete revolution at each operation of the machine, and the shaft 71, through the train of gears 72 and gears 74, is given a similar movement. The shaft 70 also has fast upon it a ratchet 36 with which co-operates 10 a pawl, to prevent reverse movement of the shaft, and thereby to compel a complete movement of the reciprocating rack frame in each direction, and inasmuch as the operated keys are locked to said frame by said coupling bar, 15 the provision of the ratchet 36 and co-operating pawl insures a complete stroke of an operated key in each direction after it has been started, as in the machine described in my prior application. As seen in Fig. 4 the shaft 20 71 has secured upon it at its extreme left hand end, outside the frame and plate W', two cams B4 and B5, the former of which is adapted to co-operate with an anti-friction roller C7 mounted upon the lower rear end of a link  $\mathrm{C}^3$ 25 which is pivoted near said end to the lower forward end of an arm C5 loosely hung to a support upon the plate W' at C. The upper forward end of the link C3 is pivoted to an ear D4 upon a vertical arm or plate D' which 30 is pivoted at its lower end to a suitable support D<sup>2</sup> upon the plate W'. A spring D<sup>3</sup> coiled around the support D2 of the arm D' and bearing against a pin upon the plate W' presses the arm D' rearward, its movement in 35 that direction being limited by a stop D<sup>8</sup> upon the plate W'. Secured to a shoulder or ledge upon the inner side of the arm D' is an arm D<sup>6</sup>, in this instance composed of a series of thin spring plates, which projects rearward 40 at a right angle to the arm D' and carries at | its rear end a platen D'immediately above and adapted to co-operate with the two type wheels. When the front end of a key lever is depressed and the shaft 71 given a half 45 revolution in a forward direction its cam B4 rides under the roller C7 on the link C3 and moves the latter upward and forward, tilting the arm D' and the platen arm and platen to the positions indicated by the dotted lines, 50 against the stress of the spring D3. Just as the operated key completes its downward stroke the cam D4 clears the roller C7 and the spring D<sup>3</sup> throws the parts back to normal position, the arm D' being arrested by the stop 55  $D^8$  and the resiliency of the platen arm  $D^6$ permitting the momentum of the parts to throw the platen D<sup>7</sup> against the type wheels. A horizontal plate M, having an upwardly curved rear end, is secured at its inner edge 60 to the plate W' and extends over the type | in this instance of hopper form and having wheels. It is provided with an opening M' immediately above the type wheels, through which the platen D<sup>7</sup> may pass. The supply of paper strip N<sup>2</sup> is carried

65 upon a spool N mounted upon a stud N', whence the strip is led forward beneath a guide O4, thence beneath the platen D7, thence I

forward around a guide roller I loosely mounted on a stud I', thence downward between a pair of feed rollers G7G9, and thence around 70 a storage spool N<sup>3</sup> loosely mounted on a stud N<sup>4</sup>. The shaft 71 has fast upon it besides the cams B4 B5 a grooved wheel or disk B6, and the storage spool N³ has fast upon it a similar, though larger, grooved wheel. A yield- 75 ing cord or band N<sup>5</sup> passed around these two grooved wheels serves to transmit movement from the shaft 71 to the storage spool N3, to cause the latter to wind up the length of paper strip N<sup>2</sup> advanced to it by the feed roll- 80 ers G<sup>7</sup> G<sup>9</sup> at each operation, the yielding character of the cord or band N<sup>5</sup> permitting it to slip upon the grooved driving wheel B6 when the storage spool has been turned far enough to wind up the slack in the paper 85 strip; as will be readily understood. The feed-rollers G<sup>7</sup> G<sup>9</sup> are driven by a train of gearing from the shaft 71, the latter having fastupon it a pinion which meshes with a large gear wheel G', which latter has secured to its 90 side a pinion G<sup>2</sup> which meshes with a large gear wheel G3, which latter in turn meshes with a pinion G4 loosely mounted upon a support G<sup>5</sup> and meshing with a large gear wheel G. The latter has fastened to it the feed 95 roller G<sup>7</sup> and also a toothed wheel G<sup>8</sup>, which latter meshes with a toothed wheel G10 fast upon the feed roller G9. The roller G9 is carried by a swinging arm G12 hung to a pivotal support G14 near its upper end and pressed 100 rearward by a spring G15 to hold the roller G9 against the roller G7. An endless inking ribbon is passed around guide rollers J4, J8 and J<sup>10</sup> at three corners of a square, and between a pair of feed-rollers H, H' at the other cor- 105 ner of the square. The roller H' is fast upon the gear G<sup>3</sup> above referred to and the roller H is mounted in a swinging arm H<sup>3</sup> and pressed by a spring H10 against the roller H', and the movement imparted to the gear G<sup>3</sup> 110 from the shaft 71 causes the rollers H H' to advance the inking ribbon at each operation of the machine, to present a fresh inking surface at the printing point. The guide roller J<sup>4</sup> is carried by an arm J' which is pressed 115 by a spring J<sup>5</sup> downward and rearward to hold the inking ribbon taut. The inking ribbon may be of the ordinary character, and be removed and a new one applied whenever it has become so used or dry as to print im- 120 perfectly; but I have provided means for automatically inking the ribbon and keeping it in fresh condition. Secured to the upper end of a bracket plate L', above the inking ribbon in its passage from the feed rollers H 125 H' to the tension roller J4, is an ink well L, a series of strips of felt secured in its lower contracted end, with their lower ends depending from it and bearing against the ink- 130 ing ribbon. A supply of ink is kept in the well L and transmitted by the strips of felt to the inking ribbon as the latter is drawn across the former.

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For the purpose of automatically locking the type wheels in position while the printing is being effected I have provided a pair of locking arms or dogs E<sup>5</sup> E<sup>6</sup> pivoted at their 5 rear ends to the opposite sides of a plate E4 projecting rearward from the bracket plate A7 heretofore described, Figs. 4 and 6. These arms have secured between them, below and in front of the plate E<sup>4</sup>, a block F<sup>2</sup>, so that ro the arms and said block constitute a rigid frame hung to the plate E4. The forward ends of the arms E<sup>5</sup> E<sup>6</sup> are bent inward toward each other so that their toothed ends may co-operate with toothed locking wheels A14 secured to the hubs of the respective type wheels, these toothed wheels lying within the vertical planes of the sides of the type wheels, and the bent ends of the arms E<sup>5</sup> E<sup>6</sup> permitting their locking teeth to engage the toothed 20 wheels. The block F<sup>2</sup> has upon its lower side a tubular housing in which is confined a spring plunger F<sup>3</sup> whose lower end rests upon the periphery of the cam disk B5. During the downward stroke of an operated key and for-25 ward half of the revolution of the shaft 71 the cam B<sup>5</sup> will ride under the spring plunger F<sup>3</sup> just at the end of such half revolution and throw the arms E<sup>5</sup> E<sup>6</sup> upward and engage their teeth with the toothed wheels A<sup>14</sup>, and 30 hold the type wheels locked in their adjusted positions while the printing is being effected, after which, at the beginning of the return stroke of the operated key and second half of the revolution of the shaft 71 the cam B<sup>5</sup> will 35 clear the lower end of the spring plunger F<sup>3</sup> and the arms E<sup>5</sup> E<sup>6</sup> will drop to normal position and release the type wheels.

The supply spool N before referred to has secured to it a grooved disk or wheel with which co-operates a brake arm O pressed against it by a spring O2, to preserve the

proper tension upon the strip.

From the foregoing description it will be seen that when the front end of any key le-45 ver is depressed its rear end enters the corresponding slot in the differential plate, moves said plate the proper distance in one direction as it travels through the inclined portion of the slot, thereby turning the type wheel un-50 til the number representing the value of the key is brought to the printing point, and then during the further idle movement of the key, in the vertical upper end of the slot, the locking dogs are thrown into engagement with 55 the toothed locking wheels, to lock the operated type wheel in its adjusted position, and the platen is retracted, against the stress of its spring, and then released and thrown against the type wheels to effect the print-60 ing; and that after this has been done, during the first downward movement of the rear end of the operated key, through the vertical upper end of its slot, the locking dogs will be disengaged from the type wheels and 65 the latter released, and then during the return of the key through the inclined portion of its slot the differential plate will be re-

turned to normal position and carry the operated type wheel with it to such position, after which the rear end of the key will pass down- 70 ward through the vertical lower end of its slot and be disengaged from the differential plate. As before stated the provision for idle movement of the operated key (with respect to the differential plate) at the end of its 75 positive stroke is particularly important in my present invention, since it enables the type wheel to be turned to proper position and then come fully to rest before it is locked in such position and before the platen is act-80 nated to effect the printing. Under the construction illustrated the provision for this idle movement is made by terminating the inclined slots in vertical portions at their upper ends, but provision for such movement may 85 be made in various other ways in machines employing differentially movable plates or frames of a different character from that employed in my machine; and my invention in this respect is not restricted to the employ- 90 ment of a differentially movable plate having inclined slots, nor to an endwise moving plate or frame, or a plate or frame which slides in or upon guide ways, but, in its broader scope, contemplates, in the combinations set forth 95 in my claims, all forms and arrangements of differentially movable plates or frames employed to set a type wheel or carrier and actuated by the keys in such manner as to be given their proper movements, proportionate 100 to the values of the keys, by less than the full positive strokes of the keys, for the purpose described.

I am aware that it has heretofore been proposed to combine a series of keys of different 105 values with a differentially movable plate or frame, and a type wheel geared to said frame, and a spring-actuated platen arranged to be retracted during the positive stroke of the key and released and thrown against the type 110 wheel at the end of such stroke, to effect the printing; and that in some instances such differentially movable plate or frame has consisted of an endwise moving plate arranged to slide in guide ways and provided with a 115 series of differentially inclined slots; but in all of the instances with which I am familiar the complete positive stroke of an operated key was required to give the frame or plate the necessary movement for setting the type 120 wheel or carrier, and no provision was made for idle movement of the key, with respect to such frame or plate, at the end of the positive stroke of the key, as hereinbefore explained.

As seen in Figs. 2 and 3 the machine is provided in its lower portion with a drawer compartment in which fits a backwardly and forwardly sliding money drawer G having secured upon its opposite sides angle plates 105, 130 whose horizontal portions rest upon rollers 106 mounted upon bearings upon the inner sides of the drawer compartment. The horizontal flanges of the plates 105 also prefer-

000,000

ably travel beneath guide rollers 107 upon the J walls of the compartment, while rollers 108 mounted upon vertical axes in bearings secured to the sides of the compartment receive 5 the side thrusts of the drawer and guide it in and out without friction. The vertically reciprocating rack frame heretofore described has an extension depending into the drawer compartment and carrying a cam plate 109 ro having an inclined or beveled front edge adapted to engage the roller 110 upon a stud on the side of the drawer when the front end of a key lever is depressed and the rack frame and cam plate lifted. The drawer is auto-15 matically locked and held in its closed position by a latch 111 fast on a rock-shaft 112. This rock-shaft also has fast upon it an arm 79 which co-operates with a cam 78 fast upon the rotary shaft 71 heretofore described. 20 When the operating keys are in their normal position of rest the cam 78 projects rearward from the shaft 71 and the latch arm 111 rests upon a cross piece of the framework below and immediately in rear of the shaft 112, the 25 rear end of the latch arm in such position resting immediately in front of the upper edge of the rear wall of the drawer and holding the latter closed. When the front end of

any key lever is depressed, as in Fig. 3, the 30 first forward movement of the rotary shaft 71 will carry the cam 78 under the rear end of the arm 79 and lift the latch 111 out of engagement with the drawer, as seen in said figure, and the further depression of the front 35 end of the key lever and lifting of the rack frame and cam plate 109 will cause the latter

to force the drawer part way open.

There is combined with each set of keys a series of laterally displaceable stops 7, Fig. 2, 40 for preventing simultaneous operation of two or more keys in the same set, as is usual in this class of machines, and for the purpose of locking the unoperated keys until the operated one has been returned completely to ini-

45 tial position the keys are provided in line with the stops 7 with pendent extensions 9, Fig. 3, which co-operate with the stops in the

usual well known manner.

The indicating mechanism of the machine 50 may be either that of my aforesaid pending application, or that of another prior pending application, Serial No. 414,294, filed December 7, 1891, or it may be of any other suitable character.

Having thus fully described my invention, I claim—

1. The combination of a differentially movable plate or frame, a plurality of operating keys of different values co-operating there-60 with to move the same different degrees proportionate to the values of the keys, and each key having a movement independent of the plate at the end of its positive stroke, a type carrier actuated by the movable plate, and a 65 platen co-operating with the type carrier to

effect the printing of the value of the operated key, substantially as described.

2. The combination of a differentially movable plate or frame, a plurality of operating keys of different values co-operating there- 70 with to move the same different degrees proportionate to the values of the keys, and each key having a movement independent of the plate at the end of its positive stroke, a typecarrier actuated by the movable plate, a lock-75 ing device co-operating with the type carrier and actuated by the operated key during its independent movement at the end of its positive stroke, and a platen co-operating with the type carrier during such independent 8c movement of the key to effect the printing of the value of the latter, substantially as described.

3. The combination of a differentially movable plate or frame, a plurality of keys of dif- 85 ferent values co-operating therewith to move the same different degrees proportionate to the values of the keys, and each having a movement independent of the plate at the end of its positive stroke, a type carrier act- 9c uated by the movable plate, a platen co-operating with the type carrier, a spring for throwing the platen against the carrier to effect the printing, and means actuated by the keys for retracting the platen during the 95 positive stroke of the operated key and releasing it during the independent movement of the key at the end of such positive stroke,

substantially as described.

4. The combination of a differentially mov- 100 able plate or frame, a plurality of keys of different values co-operating therewith to move the same different degrees proportionate to the values of the keys, and each key having a movement independent of the plate at the 105 end of its positive stroke, a type carrier actuated by the movable plate, a locking device co-operating with the type carrier and actuated by the operated key during its independent movement at the end of its positive 110 stroke, a platen co-operating with the type carrier, a spring for throwing the platen against the carrier, and means actuated by the operated key for retracting the platen during the positive stroke of the key and re- 115 leasing it during the independent movement of the key at the end of such stroke, substantially as described.

5. The combination of a differentially movable plate or frame, a plurality of operating 120 keys of different values co-operating therewith to move the same different degrees proportionate to the values of the keys, a type carrier actuated by the movable plate or frame, a platen co-operating with the type carrier, 125 a rotary cam for actuating the platen, and means intermediate the keys and cam for rotating the latter, substantially as described.

6. The combination of a differentially movable plate or frame, a plurality of operating 130 keys of different values co-operating therewith to move the same different degrees proportionate to the values of the keys, a type carrier actuated by the movable plate or frame,

a platen co-operating with the carrier, a spring for throwing the platen against the carrier, a rotary cam for retracting the platen against the resistance of its spring, and means inter-5 mediate the keys and cam for rotating the

latter, substantially as described.

7. The combination of a differentially movable plate or frame, a plurality of operating keys of different values co-operating thereto with to move the same different degrees proportionate to the values of the keys, and each key having a movement independent of the plate at the end of its positive stroke, a type carrier actuated by the movable platen, a 15 platen co-operating with the carrier, a spring for throwing the platen against the carrier, a rotary cam for retracting the platen against the stress of its spring, and means intermediate the keys and cam for rotating the lat-20 ter, substantially as described.

8. The combination of a differentially movable plate or frame, a plurality of keys of different values co-operating therewith to move the same different degrees proportionate to 25 the values of the keys, a type carrier actuated by the movable plate or frame, a platen cooperating with the type-carrier, a locking device co-operating with the carrier to lock it in its adjusted position while the printing is 30 being effected, a rotary cam for actuating said locking device, and means intermediate the keys and cam for rotating the latter, substan-

tially as described.

9. The combination of a differentially mov-35 able plate or frame, a plurality of operating keys of different values co-operating therewith to move the same different degrees proportionate to the values of the keys, a type carrier co-operating with the movable plate 40 or frame, a platen co-operating with the type carrier, a spring for throwing the platen against the carrier, a rotary cam for retracting the platen against the stress of the spring. a locking device co-operating with the carrier 45 to lock it in adjusted position while the printing is being effected, a rotary cam for actuating said locking device, and means intermediate the keys and cams for rotating the latter, substantially as described.

10. The combination of a differentially movable plate or frame, a plurality of keys of different values co-operating therewith to move the same different degrees proportionate to the values of the keys, and each key having 55 a movement independent of the plate at the end of its positive stroke, a type carrier actuated by the movable plate or frame, a platen co-operating with the carrier, a spring for throwing the platen against the carrier, a ro-60 tary cam for retracting the platen against the stress of its spring during the positive stroke of a key and releasing it during the independent movement of the key at the end of such stroke, a locking device co-operating with the 65 type-carrier to lock it in adjusted position while the printing is being effected, a rotary

effect the locking of the type carrier during the independent movement of the key at the end of its positive stroke, and means inter- 7° mediate the keys and cams for rotating the latter, substantially as described.

11. The combination of a plurality of operating keys of different values, a differentially movable plate or frame actuated thereby and 75 movable different degrees by different keys proportionate to their values, a type carrier actuated by such movable plate or frame, a platen co-operating with the carrier, a pair of feed rollers for moving a paper strip between 80 the platen and type carrier, a rotary shaft for actuating the feed rollers to advance the strip, and means intermediate the keys and shaft for rotating the latter, substantially as de-

scribed.

12. The combination of a plurality of operating keys, a differentially movable plate or frame actuated thereby and movable different degrees by different keys proportion at eto their values, a type carrier actuated by the movable 90 plate or frame, a platen co-operating with the carrier, an inking ribbon led between the platen and carrier, a pair of feed rollers for advancing the ribbon, a rotary shaft for actuating the feed rollers, and means intermediate the 95 keys and shaft for actuating the latter, substantially as described.

13. The combination of a plurality of operating keys of different values, a differentially movable plate or frame actuated thereby and roc movable different degrees by different keys proportionate to their values, a type carrier actuated by such movable plate or frame, a platen co-operating with the carrier, an inking ribbon led between the platen and carrier, 105 means actuating by the keys for automatically advancing said ribbon, and an ink well having a distributing pad or brush bearing against the inking ribbon and operating to automatically ink the latter, substantially as IIC described.

14. The combination of a plurality of keys, a movable plate provided with a plurality of differentially inclined slots, each terminating at one end in a portion in line with the move- 115 ment of the key, a type carrier actuated by such plate, and a platen co-operating with the

carrier, substantially as described. 15. The combination of a plurality of keys, a movable plate provided with a plurality of 120 differentially inclined slots, each terminating at one end in a portion in line with the movement of the key, a type carrier actuated by such plate, a platen co-operating with the carrier, a spring for throwing the platen against 125 the carrier, and means actuated by the keys for retracting the platen against the stress of its spring, substantially as described.

16. The combination of a plurality of keys of different values, a differentially movable 130 plate or frame actuated thereby, the differential movements of the plate being effected by a series of differential cam engaging surcam co-operating with such locking device to I faces between the keys and plate, each of said

surfaces beginning and ending in portions in line with the strokes of the keys, a type carrier actuated by such movable plate, and a platen co-operating with the carrier, substantially as described.

5 tially as described.

17. The combination of a plurality of keys of different values, a differentially movable plate or frame actuated thereby, said plate or frame being provided with a plurality of differential cam surfaces adapted to be engaged by the different keys and each beginning and ending in a portion in line with the stroke of the key, a type carrier actuated by such movable plate, and a platen co-operating with the carrier, substantially as described.

18. The combination of a plurality of keys, a movable plate provided with a plurality of differentially inclined slots, each of said slots terminating at both ends in portions in line with the movement of the key, a type carrier actuated by such movable plate, and a platen co-operating with the carrier, substantially

as described.

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19. The combination of a plurality of operating key levers, a vertically arranged longitudinally movable plate extending transversely across the rear ends of said levers and provided with a plurality of differentially inclined slots, each terminating in a vertical portion at its upper end, a type carrier actuated by such plate, a platen co-operating with the carrier, a spring for throwing the platen

against the carrier, and means actuated by the keys for retracting the platen against the stress of the spring, substantially as de- 35 scribed.

20. The combination of a plurality of operating keys of different values, a differentially movable plate or frame actuated thereby and movable different degrees proportionate to 40 the values of the keys, a type carrier actuated by such movable plate or frame, a platen cooperating with the carrier, a spring for throwing the platen against the carrier, a rotary cam for retracting the platen against the 45 stress of its spring, a locking device for locking the carrier in adjusted position while the printing is being effected, a rotary cam for actuating such locking device, a pair of feed rollers for drawing a paper strip between the 50 platen and carrier, a pair of feed rollers for advancing the ribbon, a rotary shaft for driving the feed rollers and the rotary cams, and means intermediate the keys and shaft for rotating the latter, substantially as described. 55

In testimony whereof I have signed my name to this specification, in the presence of two subscribing witnesses, on this 23d day of

February, A. D. 1892.

WILLIAM KOCH.

Witnesses:
GEORGE WINTER,
EDW. L. WATERBURY.

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