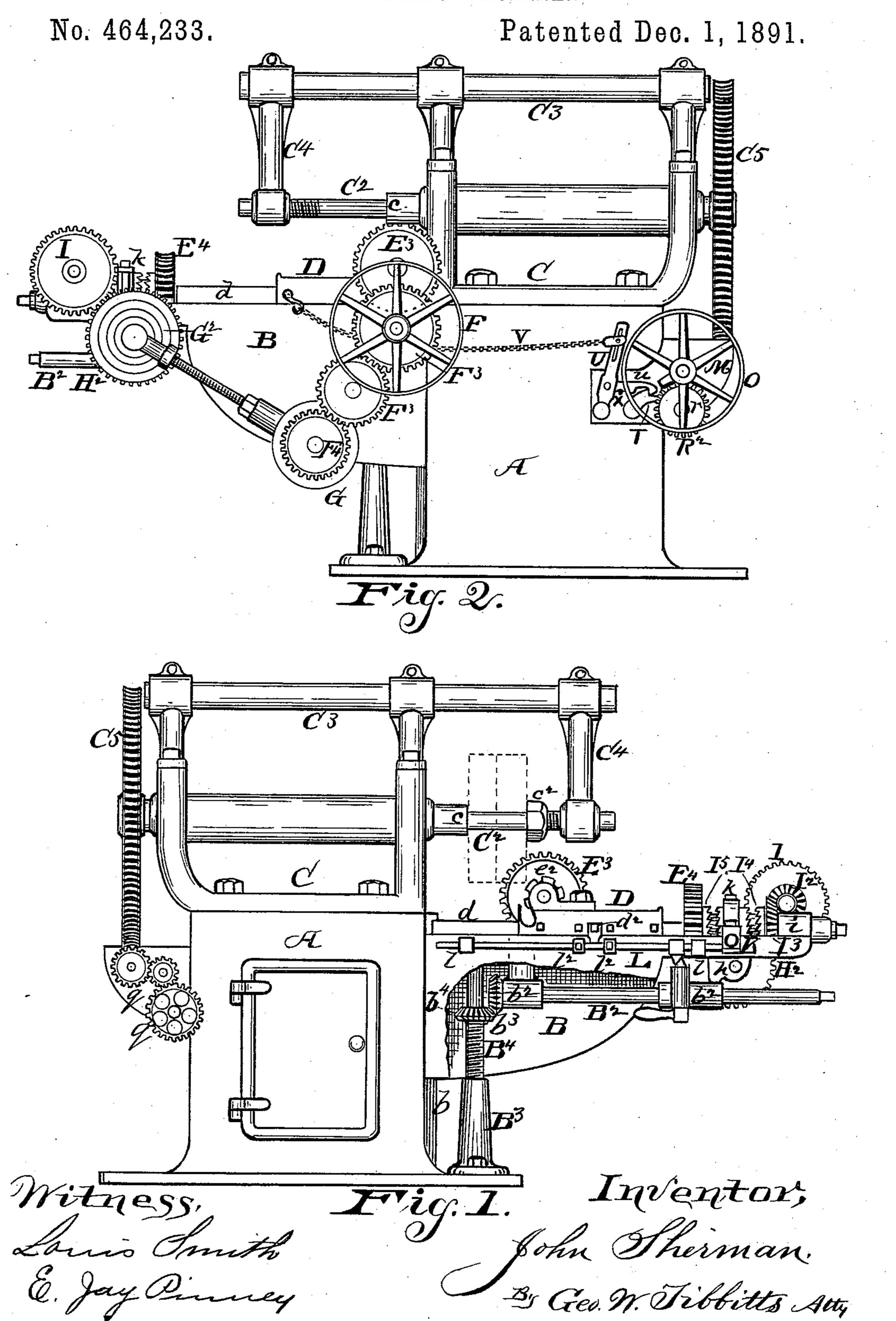
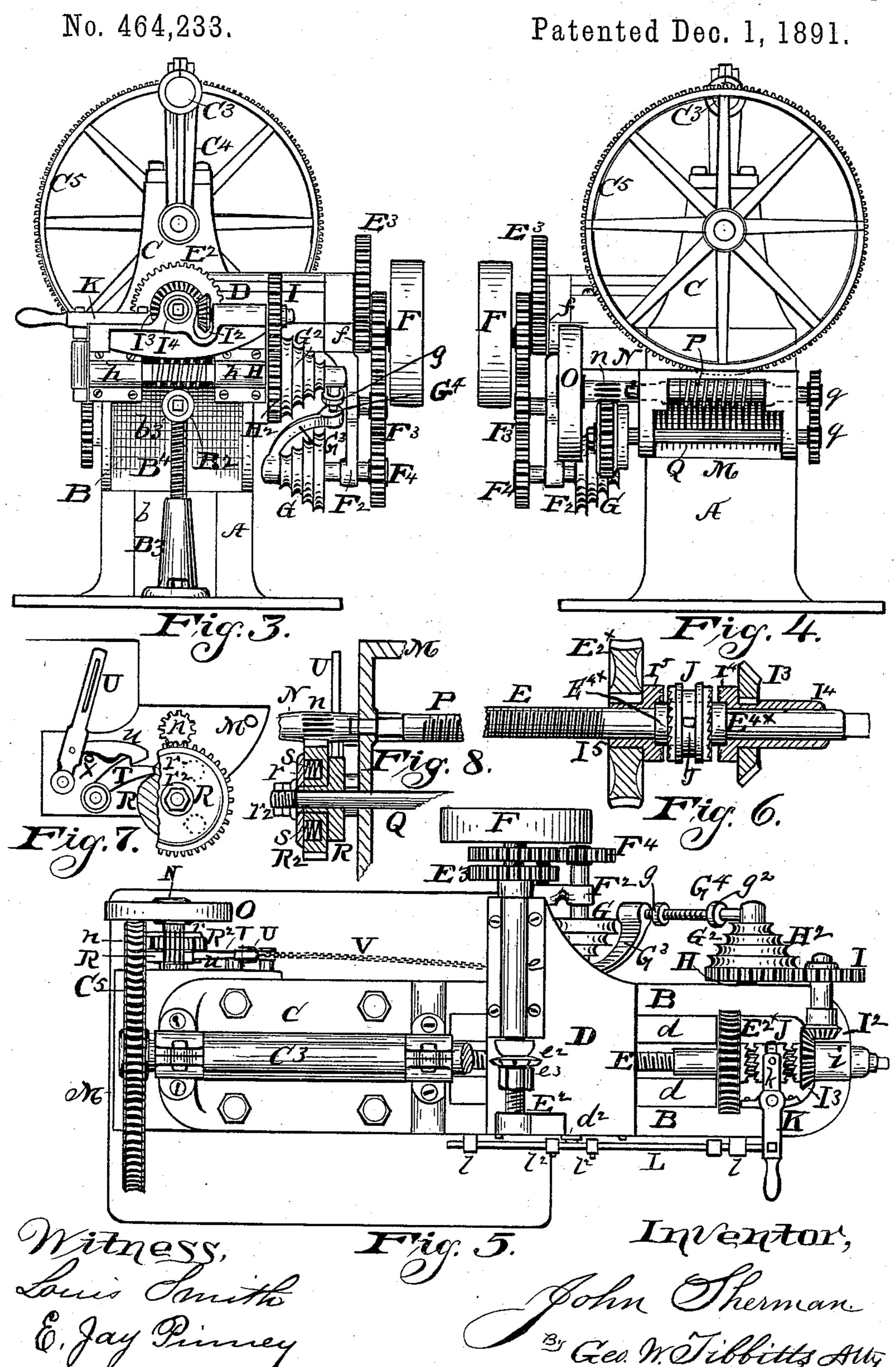
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United States Patent Office.

JOHN SHERMAN, OF CLEVELAND, OHIO, ASSIGNOR OF ONE-HALF TO LOUIS SMITH, OF SAME PLACE.

GEAR-CUTTING MACHINE.

SPECIFICATION forming part of Letters Patent No. 464,233, dated December 1, 1891.

Application filed July 13, 1891. Serial No. 399,296. (No model.)

To all whom it may concern:

Be it known that I, John Sherman, a citizen of the United States, and a resident of Cleveland, in the county of Cuyahoga and State of Ohio, have invented certain new and useful Improvements in Gear-Cutting Machines, of which the following is a specification.

This invention relates to machines for cutto ting gear-teeth; and it consists in the novel construction and combinations of parts, as hereinafter fully described, and pointed out in the claims.

In the accompanying drawings, Figure 1 is a front side elevation. Fig. 2 is a reverse side elevation. Fig. 3 is a front end elevation. Fig. 4 is a rear end elevation. Fig. 5 is a top or plan view. Fig. 6 is a sectional view of the clutch. Fig. 7 is a side view of a latch-stop mechanism. Fig. 8 is a transverse sectional view of same.

A represents a standard and base, which supports all the working parts of the machine.

B is a vertically-adjustable table attached to one end of said standard by means of a dovetail groove and slide b. To the under side of the table is provided a horizontal shaft B², fixed to turn in hangers b², to the inner end of which is attached a bevel-gear b³.

B³ is a hollow internally-screw-threaded post secured to the base flange of support A, and B4 is a vertical screw playing in said post, whose upper end is fixed in a socket in the 35 under side of the table. It is also provided with a bevel-gear b^4 , meshing with the aforesaid gear b^3 . By means of a crank on the outer end of shaft B2 the said table may be elevated or depressed for adjustment to ac-40 commodate it to the size of wheel being cut. To the top of the standard a frame C is attached provided with a work-holding shaft C2, and above this is provided a parallel bar C³, provided with a removable hanger C4, em-45 ployed for supporting the end of the shaft C2, projecting over the table B. The said shaft C² is employed for holding the wheels to be cut, as shown in dotted lines in Fig. 1, the shaft having a shoulder c, against which the 50 blank wheels rest and are held by a nut c^2 on the shaft. The rear end of the shaft C2 is I

provided with a large worm gear-wheel C5, referred to later on. Upon the table B is provided a sliding tool-holding bed D, having a dovetail groove riding on dovetail slides $d\ d$ 55 on said table. The bed D is made to travel back and forth by means of a screw E, playing in a nut on the under side of said bed D. On the bed D is provided a tool-holding shaft E^2 , set in a suitable box-bearing e, the shaft 60 having an arbor for holding the cutting-tool e^2 with a nut e^3 . On the outer end of toolshaft is provided a gear-wheel E³. Beneath the tool-shaft is provided a second shaft (not visible) set in suitable bearings and having 65 a pinion f meshing with said wheel E^3 , and upon which is fixed the driving-pulley F. Upon the said invisible shaft is also suspended a swinging arm F², carrying a train of gear-wheels F³ F⁴. On the shaft of the lower 70 gear is provided a cone-pulley G. Near the outer end of the table B is provided a shaft H, journaled in bearings h h, and it has a wormscrew, which turns the screw E through the medium of the worm-gear E⁴ on said screw E, 75 by which the traversing bed D is propelled. On the said shaft H is also placed a gearwheel H' and also a cone-pulley G2, to be connected by belt with pulley G for transmitting motion thereto.

G³ is a quadrant-arm connected with the shaft of lower-cone pulley, and it is also connected by a screw-rod G4 with the shaft of upper-cone pulley G². Said screw-rod is also provided with jam-nuts $g g^2$ for fastening the 85 rod in the sockets on the shaft and quadrant, by which also adjustment may be made for giving proper tension to the belt on the cone pulleys. Upon the top corner of table B is also provided a short shaft having a gear I 90 meshing with the gear H², and on its opposite end is provided a bevel pinion-gear I², which meshes with a bevel-gear I³ on a screw I⁴, Fig. 6, loosely placed on the outer end of screwshaft E and within the bearing i on the end 95 of the table B. The clutch mechanism on said shaft E comprises one of the novel features of my invention, described as follows: The said screw-shaft E has an enlargement E^{4×}, Fig. 6, forming shoulders, against which roc sleeves 14 and 15 bear, having clutch-teeth, and upon said sleeves are keyed the gears

E^{2×} and the bevel-gear I³. Upon the enlargement E4× is placed a sliding clutch-sleeve J, secured by a pin j entering a slot in the shaft, which allows it to have short longitudinal 5 movement while rotating with the shaft. A ring is fitted to the groove in the periphery of the sleeve J and is pivoted in the yoke k on the lever K, which lever is fulcrumed onto the table B. On the side of the traversing ro bed D is fixed a push pin or arm d^2 , and on the side of the table B is attached a sliding rod L, set to slide in bearings l l on the side of the table. This rod is pivotally attached to the clutch-lever K for actuating the same, 15 as hereinafter shown. On the rod L are fixed adjustable stops l² l², against which the arm d^2 strikes in the movements of the bed. On the rear end of the standard A is attached a bracket M, supporting a worm-screw mech-20 anism for intermittently actuating the feedwheel C⁵, by means of which the blank wheel being cut is turned for cutting the teeth in succession. This comprises the second novel feature of my invention and is described as 25 follows.

N is a shaft set in bearings in the bracket M, having pinion-teeth n, and is provided

with driving-pulley O.

Pisa worm-screwshaft, also having its bear-30 ings in the said bracket, and lies parallel with shaft N, with its worm-screw in mesh with the

teeth of large feed-wheel C⁵.

Q is a third shaft journaled below the shaft N, and is connected by gear g g with the 35 worm-screw shaft P. On the other end of shaft Q is provided a friction-gear mechanism by which an intermittent movement is applied to the said feed-wheel C⁵. R is a wheel upon said shaft Q, having a sleeve upon 40 which is placed a gear-wheel R² loosely, so that it may turn thereon. In one face of said wheel R² are made recesses in which are placed springs s s, bearing against a disk r on the outer end of the shaft; held by jam-nuts r^2 ,

45 by which also the tension of the springs may be regulated. In one side of the wheel R is made a notch, (seen in Fig. 7,) with which a pawl T engages to prevent the wheel and its

shaft from turning.

U is a lever pivoted by the side of the pawl and provided with a hook u, engaging with a shoulder or lug on the pawl. The lever U is connected by a chain V with the traversing tool-bed D, by means of which, when the said 55 tool-bed has moved back in its return movements, will pull on the lever U and lift the pawl out of the notch in the wheel R. Then the said wheel, being free to move, will be turned by the friction-bearing of the wheel

60 R² against it, and thereby turn the shaft Q, and through it and the gears qq impart a mo-

tion to the worm-shaft, and thus move the feed-wheel C⁵. A pin or lug X is provided on the side support, which as the lever U is pulled over causes the hook u to be raised, it hav- 65 ing an incline on its under side, which rides on the said pin or lug for the purpose of raising it. This releases the hook from the pawl and lets it drop back again, so as to catch in the notch again and stop the revolution of 70 the said wheel and shaft until the aforesaid tool-bed again makes its return movement, which will repeat this operation.

Having described my invention, I claim— 1. The combination, with screw E, having 75enlargement $E^{4\times}$, of loose clutch-sleeves I⁴ I⁵, worm-gear $E^{2\times}$, keyed onto said sleeve I^4 , wormscrew shaft H, meshing with said gear $E^{2\times}$, and cone-pulley G² for giving rotary motion to said screw E, the cone-pulley G and the 80 quadrant-arm connecting the shafts of the said pulleys, substantially as and for the pur-

pose specified.

2. The combination of screw E, having enlargement E^{4×}, loose clutch-sleeves I⁴ I⁵, gear 85 E^{2×}, and bevel-gear I³, keyed onto said sleeves, clutch-sleeve J, mounted on said enlargement, and means for shifting said clutch-sleeve, worm-screw shaft H, and the connected conepulleys, gear H², gear and short shaft I, and 90 bevel-pinion meshing with said bevel-gear I³, arranged to operate in the manner and for the purpose specified.

3. The combination, with the feed-wheel C⁵ and worm-screw shaft P, of shaft Q, each jour- 95 naled in the bracket M and connected to revolve in conjunction by gears q q, wheel R, mounted on said shaft Q and having the friction gear-wheel R, mounted on the hub of said wheel R, said gear having recesses and 100 springs s s, bearing-plate r, jam-nuts r^2 , the shaft N, having gear-teeth n meshing with said friction-gear R², and pulley O, mounted on said shaft N, all arranged to operate substantially as and for the purpose specified.

4. The combination, with feed-wheel C⁵ and worm-shaft P, of shaft Q, each journaled in the bracket M and connected to revolve in conjunction by gear qq, wheel R, mounted on shaft Q and having a notch, of the stop- 110 pawl T, lever U, having hook u, and means for operating said lever, hook, and pawl, substantially as described, for releasing said wheel R, whereby intermittent revolving movements are imparted to said feed-wheel 115 C⁵ through the medium of the friction-gear R², substantially as and for the purpose specified.

JOHN SHERMAN.

Witnesses:

GEO. W. TIBBITTS, Louis Smith.