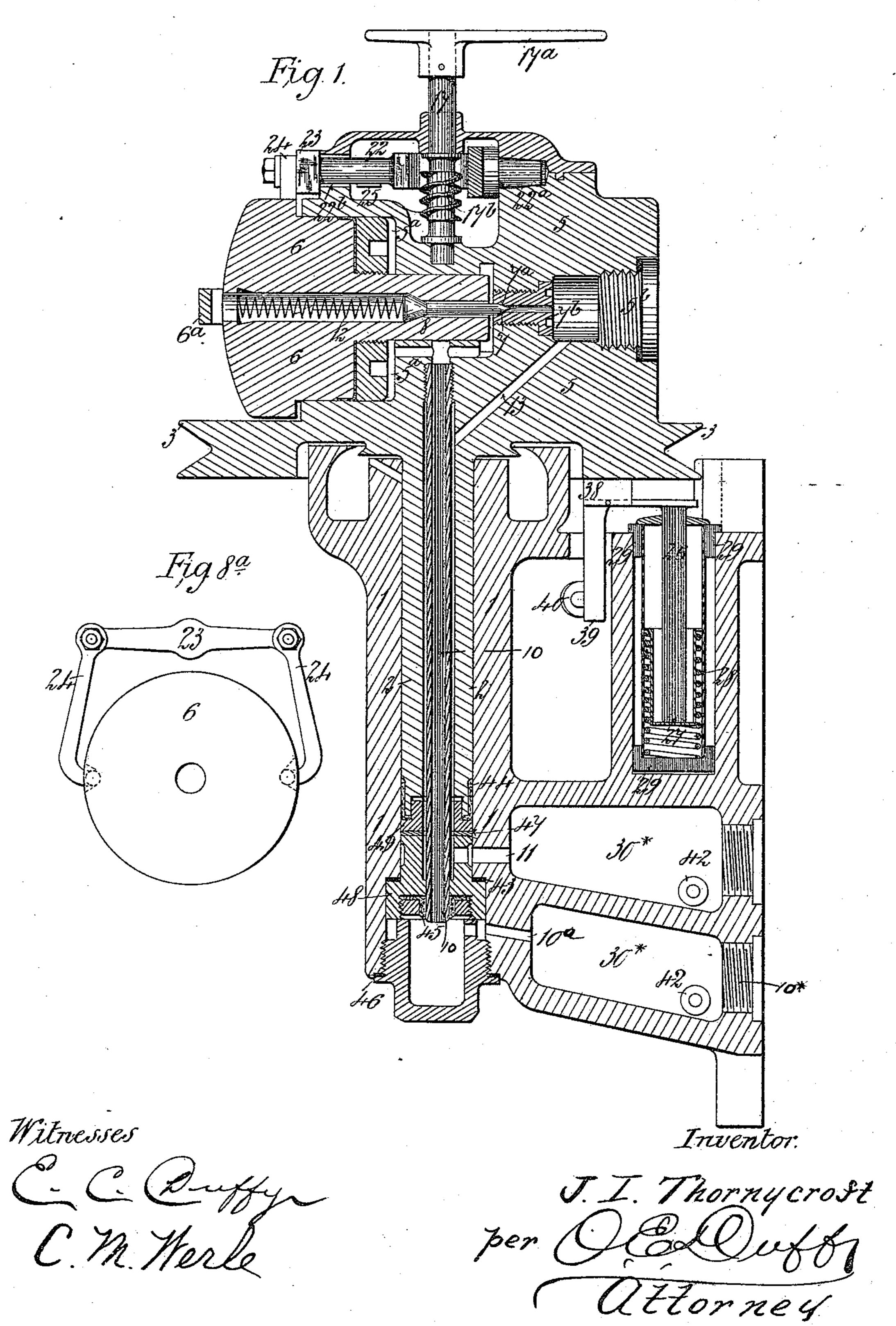
(No Model.)

3 Sheets—Sheet 1.

J. I. THORNYCROFT. SPEED INDICATOR FOR ROTATING SHAFTS.

No. 442,554.

Patented Dec. 9, 1890.

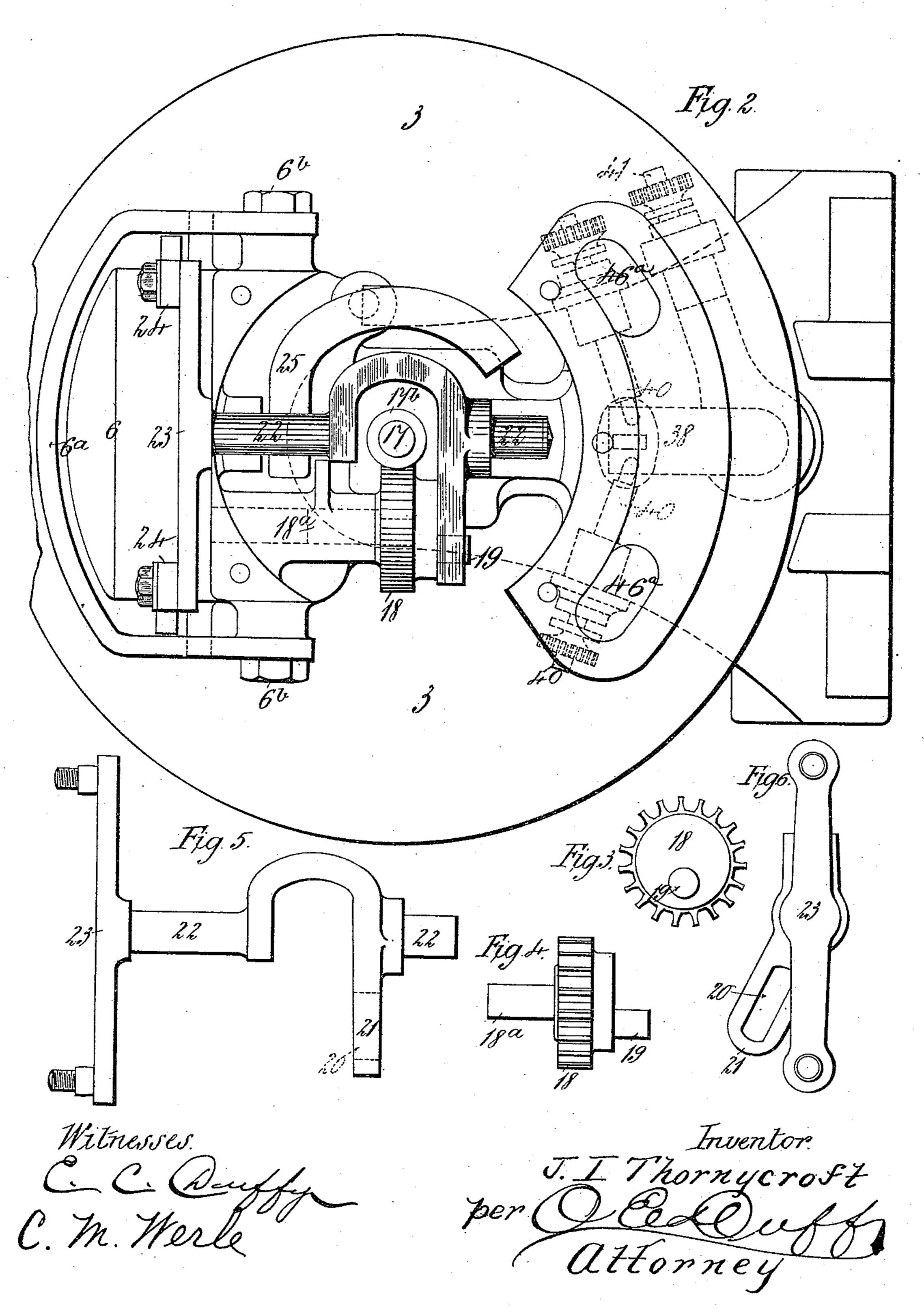


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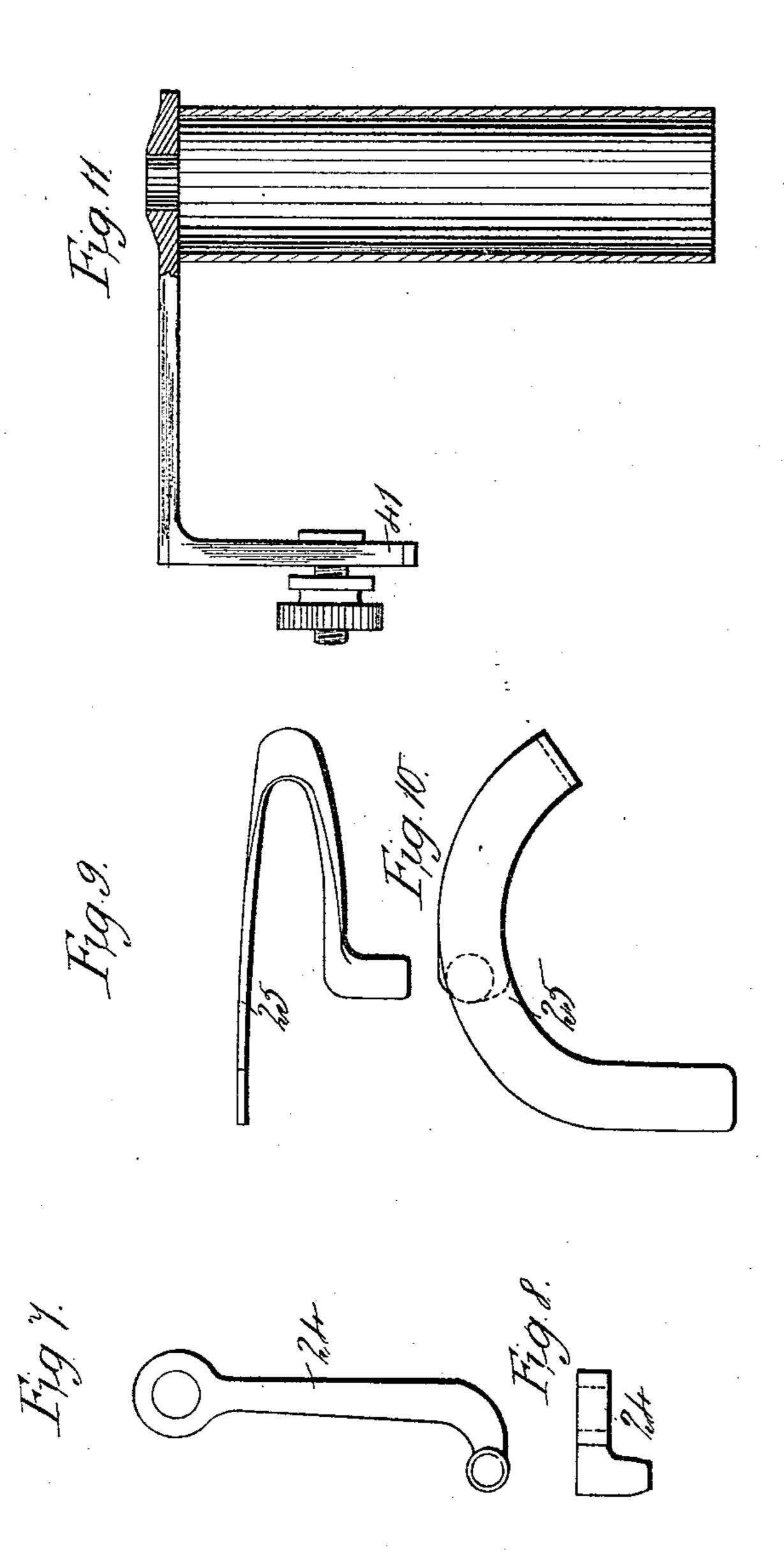
3 Sheets—Sheet 3.

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SPEED INDICATOR FOR ROTATING SHAFTS.

No. 442,554.

Patented Dec. 9, 1890.



Witnesses C. C. Duffy C. Merle

J.I. Thornycrost per Obselvefy Attorney

THE NORRIS PETERS CO., PHOTO-LITHO., WASHINGTON, O. C.

United States Patent Office.

JOHN ISAAC THORNYCROFT, OF CHISWICK, ENGLAND.

SPEED-INDICATOR FOR ROTATING SHAFTS.

SPECIFICATION forming part of Letters Patent No. 442,554, dated December 9, 1890.

Application filed February 24, 1890. Serial No. 341,635. (No model.)

To all whom it may concern:

Be it known that I, JOHN ISAAC THORNY-CROFT, a subject of the Queen of Great Britain and Ireland, residing at Chiswick, in the 5 county of Middlesex, England, have invented Improvements in Apparatus for Controlling Pressure Applicable to Indicating the Speed of Rotating Shafts, of which the following is a specification.

This invention has reference to apparatus for controlling pressure, and which is applicable for indicating the speed of rotating shafts; and the particular kind of apparatus to which my present improvements relate is that de-15 scribed in the specification of a former patent granted to me under date the 20th day of

August, 1889, No. 409,524.

The essential features of apparatus of the kind referred to are (when the pressure to be 20 controlled is less than that of the atmosphere) a hollow revolving body or valve-case (hereinafter called a "valve-case") connected | at one part to an apparatus capable of exerting a constant exhausting action, at a second 25 part to the pipe or vessel in which the pressure is to be controlled, and at a third part to the atmosphere. The first and third connections are each provided with a valve, the two valves being on one spindle and moving to-30 gether. The third inlet or opening to the valve-case is located in a plunger, piston, or diaphragm, subject to centrifugal force. It is this force which determines the pressure maintained in the valve-case and in the pipe 35 or vessel connected with it. If the motion which causes this centrifugal force is derived from the motion of a rotating shaft then the pressure in the exhausted pipe or vessel will be a function of the speed of the rotating 40 shaft, and may be made to operate a gage or gages upon which the speed of the shaft may be read; and in order that my present improvements in the above-mentioned apparatus may be clearly understood, I will, in refer-45 ring to certain parts of the original arrangement, distinguish them by the reference-numerals with which they are described in the

before-mentioned patent.

drawings annexed to the specification of my

apparatus more efficient and reliable, I dispense with the dish or cup 4, heretofore used for carrying the valve-case 5, and construct the apparatus with a combined valve-case, cylinder for plunger, and driving-spindle, all 55 formed in one piece and provided with a driving-pulley that may be of the same piece or a separate part, the arrangement being such as to dispense with the packing-ring 16 and with the packing-ring applied between the valve- 60 case and the top of the tube 10 in my original apparatus; also, I make the valve-seat in the valve-case separate from the main casting, for convenience of manufacture, and to permit of its being easily renewed.

For the purpose of reducing the friction of the plunger in a longitudinal direction and to make the same more sensitive to changes of pressure, I provide means for imparting a constant rocking motion to it. I likewise pro- 70 vide a spring, whereby the weight of the plunger is supported to keep it exactly central in its cylinder; and in this way, also, to reduce its friction, I provide a fixed safety-band to prevent the plunger accidentally flying out 75

of its cylinder.

In order that the apparatus shall indicate the direction of rotation as well as the speed, I arrange a movable electric contact or terminal in rubbing contact with a moving part of 80 the apparatus, so that the said contact or terminal is pressed against one of two fixed electric contacts or terminals. One of these contacts is fixed at each side of a movable contact and connects with an indicator which 85 shows the direction of motion.

To facilitate the comprehension of my present improvements, reference is made to the accompanying sheets of drawings, in which—

Figure 1 is a vertical central section of the 90 improved apparatus. Fig. 2 is a plan. Figs. 3 to 8 show details of the rocking apparatus. Fig. 8a is an end view of the plunger and the links and lever for imparting a to-and-fro rotary motion thereto. Figs. 9 and 10 show the 95 spring. Fig. 11 is a detail of the electric device.

5 is a valve-case formed with a bored part 5^a, constituting a cylinder, within which works To simplify the construction and render the la plunger 6 capable of endwise movement. 100

This valve-case and cylinder is, for the purposes hereinbefore mentioned, made in one piece with the vertical driving-spindle 2, which is mounted to rotate within a fixed 5 bearing 1. The driving-pulley 3 is, in the example shown, also made in one piece with the driving-spindle. A central hole is bored through the plunger, and at one part of it there is formed a valve-seat, against which to there normally rests the valve 8. Against one side of this valve 8 bears one end of a spring 12, the other end of which bears against a shoulder in the plunger 6. This spring should be so constructed and arranged as to 15 exert only a small force against the valve S. The valve 8 is connected by a rod or stem to a second valve 7, which normally rests against a seat 7^a, formed in a removable bush 7^b, which is screwed into a recess prepared for 20 it in the valve-case 5. This opening through which the bush is introduced is closed by a plug 5^b. The plunger 6 is capable of a small endwise movement in either direction from the position in which it is shown. To prevent 25 it being accidentally thrown out of its cylinder by the centrifugal force, it is partly surrounded by a safety band or bridle 6a, which is attached to the valve-case by screws 6^b 6^b. The plunger 6 receives a to-and-fro oscillat-30 ing motion to reduce its friction in its cylinder. This motion is imparted as follows: In the axial line of the rotary valve-case is a spindle 17, held stationary by an arm 17^a, which is attached to some fixed object. On

35 the spindle 17 is a worm 17^b, with which there engages a worm-wheel 18 on the spindle 18a. In the face of the worm-wheel is a crank-pin 19, which works in a slot 20 in a lever 21 formed in one with the shaft 22. On the ex-40 tremity of the shaft is a double-armed lever 23, to each end of which is pivoted a link 24. Each of these links is pivoted at its other extremity to the plunger 6. (See Fig. 8a.) The links are so pivoted to the lever and to the 45 plunger as to permit, when necessary, of a small movement of the plunger in the direction of its axis without its becoming disconnected from the lower ends of the links. For this purpose the bearing-surfaces between the 50 links and the lever and plunger may be made slightly spherical or barrel-shaped. When the valve-case is in rotation, the worm-wheel

revolves about the room 17b, and at the same time it rotates on its axis. The crank-pin 55 working in the slot 20 gives a to-and-fro oscillating motion to the shaft 22, and this motion is transferred to the plunger 6. It is well known that if a plunger or piston is kept in rotary motion it can be moved endwise with 60 little friction, and, further, that it is prevented | tion between the fixed plug 48 and a collar 49 from sticking or becoming fast in its cylinder. The spindle 22 at its inner end runs in

a bearing 22^a, and near its outer end, which works through a guide 22^b, is supported by a 65 spring 25, which keeps the links 24 always in tension, in such a manner as to take the weight of the plunger 6 and thereby reduce its fric-

tion in the cylinder. For this purpose the bearing 22° is made conical or sufficiently large to permit of the outer end of the spindle 70 making, when necessary, a slight upward turning movement about the bearing 22° as a center.

In order that the direction of motion, as well as the rate of revolution, may be indi- 75 cated, an electrical device is added to the apparatus. Within the framing there is mounted a spindle 26, standing on a footstep 27, constantly pressed upward by a spring 28. This spring is soldered to its abutment at its end, 80 as it also serves for an electric conductor. This spindle is insulated from the frame by non-conducting packings 29. Connected to the spindle 26 is an arm 38, covered or partially covered with insulating material. This 85 arm is kept in rubbing contact with the under side of the pulley 3 by the spring 28. When the pulley revolves in one direction it carries the arm with it as far as it is permitted to go, and when it revolves in the other 90 direction it does similarly. From the arm 38 there depends a finger 39, capable of moving within the limits defined by two stops 40 40. These stops are the terminals of two electric circuits, which have a common return through 95 the terminal 41, which is in metallic contact with the footstep-bearing of the spindle 26, through the spring 28. If the two terminals 40 40 be connected to the two poles of an electric battery, and if the terminal 41 be 100 connected to the center of the same battery, then the current will flow through the conductor connecting the terminal 41 to the battery in one direction when the pulley 3 rotates to the right and in the opposite direc- 105 tion when it rotates to the left. An instrument such as a galvanoscope interposed in the circuit will thus show the direction of motion of the pulley.

The connections to the pipe in which the 110 pressure is to be controlled and to the exhausting apparatus now form part of the frame at 10* and 11, respectively. Cavities 30* are formed leading to these orifices, and are employed to catch the lubricant which 115 leaks down the spindle. This can be drawn off by cocks at 42 42. To prevent communication between the pipe in which the pressure is to be controlled, the exhausting apparatus and the atmosphere packing is employed 120 around the spindle 2. This consists of a leather ring 43 and a cupped leather 44, and a smaller cupped leather 45.

46 is a packing-ring to prevent entry of air to the lower cavity 30*.

47 is a steel ring that serves to lessen fricthat may revolve with the spindle 2, and which serves to hold the packing-ring 44 in place.

130

In order to prevent variation in the velocity ratio of the driving-pulley (the velocity of which is to be indicated) and the pulley 3 driven thereby should the driving-cord be-

come reduced in diameter due to wear or other cause, the angle of the V-shaped groove of each pulley should be such that the distances between the center of the cord and the bot-5 toms of the grooves of the two pulleys when the cord is in place in the grooves will be in the ratio of the diameters of the two pulleys, measuring from the bottoms of the grooves.

To enable the revolving part to be readily 10 balanced, holes 46a 46a are cast in it for the

reception of lead weights.

Let it be assumed that the spindle is rotating with a uniform velocity and that the exhausting apparatus is in operation. The 15 plunger 6 will be urged outward by centrifugal force, and this force will be balanced by the excess of pressure on its outer end over that on its inner end. Any outwardly-acting centrifugal force in the coupled valves 7 and 20 8 is resisted by the spring 12. Now should the speed of rotation of the spindle increase, the plunger will move outward and the valve 7 will leave its seat. Air will then be drawn from the external indicator-tube through the 25 tube connected to the frame at 10*, the passage 10^a, and the tube 10 past the valve 7 into the passage 13 and down the annular passage in the spindle to the chamber at the bottom of it, and thence by the outlet 11 to the ex-30 hausting apparatus. When the pressure has been reduced sufficiently to balance the centrifugal force of the plunger, the latter returns to its former position and the valve 7 closes the passage 13. Thus the pressure in 35 the external indicator-tube, on which is or are mounted one or more vacuum-gages, will vary with the speed of revolution of the shaft from which the pulley 3 is driven, and this speed will be indicated by the said gage or gages, as 40 in the apparatus described in the said former specification. Should the speed of the shaft be decreased the plunger will move inward and the external air will pass the valve 8 until the partial vacuum existing below the 45 plunger and in the indicator-pipe is so far reduced that the plunger moves back to its normal position and closes the valve 8.

What I claim is—

1. In apparatus for indicating the speed of 50 rotating shafts, a rotating valve-case with plunger-cylinder formed in one with the spindle on which it is mounted, and connected at one part to an apparatus for increasing or reducing fluid-pressure, such as an exhauster, 55 at another part to the pipe in which pressure is to be varied, and at a third part to a source of constant pressure, such as the atmosphere, in combination with a body, such as a plunger, arranged to rotate with said valve-case 60 and itself subjected to the direct action of centrifugal force, and a valve or valves controlling two of the connections to the said valve-case, substantially as described, for the purposes specified.

2. In apparatus for indicating the speed of rotating shafts, a rotating valve-case with plunger-cylinder formed in one with the spin-

dle on which it is mounted and connected at one part to an apparatus for increasing or reducing fluid-pressure, such as an exhauster, 70 at another part to the pipe in which pressure is to be varied, and at a third part to a source of constant pressure, such as the atmosphere, a plunger arranged to be rotated with said valve-case and to be subjected to the direct 75 action of centrifugal force, and two valves, of which one is seated in the said plunger, substantially as described, for the purpose

specified.

3. In apparatus for indicating the speed of 80 rotating shafts, a rotating valve-case formed in one with the spindle on which it is mounted, and connected at one part to an apparatus for increasing or reducing fluid-pressure, such as an exhauster, at another part to the pipe 85 in which pressure is to be varied, and at a third part to a source of constant pressure, such as the atmosphere, a plunger arranged to be rotated with said valve-case and to be subjected to the direct action of centrifu- 90 gal force, and two valves, of which one is seated in the said plunger, while the other commands a connection between the apparatus for increasing or reducing fluid-pressure and the space in which the pressure is to 95 be varied, substantially as described.

4. In apparatus for indicating the speed of rotating shafts, the combination, with a rotating valve-case with plunger-cylinder and a plunger subject to centrifugal force, of a 100 safety-band arranged to prevent the plunger accidentally flying out of said cylinder, sub-

stantially as described.

5. In apparatus for indicating the speed of rotating shafts, a plunger having a con- 105 stant oscillating motion and subject to centrifugal force, to a fluid-pressure which is capable of being varied, and to a constant fluidpressure, such as the atmosphere, substantially as herein described, for the purposes 110 set forth.

6. In apparatus for indicating the speed of rotating shafts, the combination, with a plunger subject to centrifugal force, to a fluidpressure which is capable of being varied, 115 and to a constant fluid-pressure, such as the atmosphere, of the links 24, lever 23, spindle 22, with slotted arm 21, worm-wheel 18, with crank-pin 19, and fixed worm 17^b, substantially as described, for the purpose set forth. 120

7. In apparatus for indicating the speed of rotating shafts, the combination, with a plunger subject to centrifugal force, to a fluidpressure which is capable of being varied, and to a constant fluid-pressure, such as the 125 atmosphere, of the links 24, lever 23, spindle 22, with slotted arm 21, worm-wheel 18, with crank-pin 19, fixed worm 17^b, and spring 25, arranged to support said plunger, substantially as described, for the purpose set forth. 130

8. In apparatus for indicating the speed of rotating shafts, the combination, with a driving-pulley 3, of a spindle supported by a footstep constantly pressed upward by a spring,

an arm carried by said spindle and in rubbing contact with, but electrically insulated from, said pulley, a movable electric terminal carried by said arm, two fixed terminals between which said terminal can move, and electrical connections, substantially as herein described, for the purpose set forth.

In testimony whereof I have signed my

name to this specification in the presence of two subscribing witnesses.

JOHN ISAAC THORNYCROFT.

Witnesses:

W. Cross,
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WM. THOS. MARSHALL,

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