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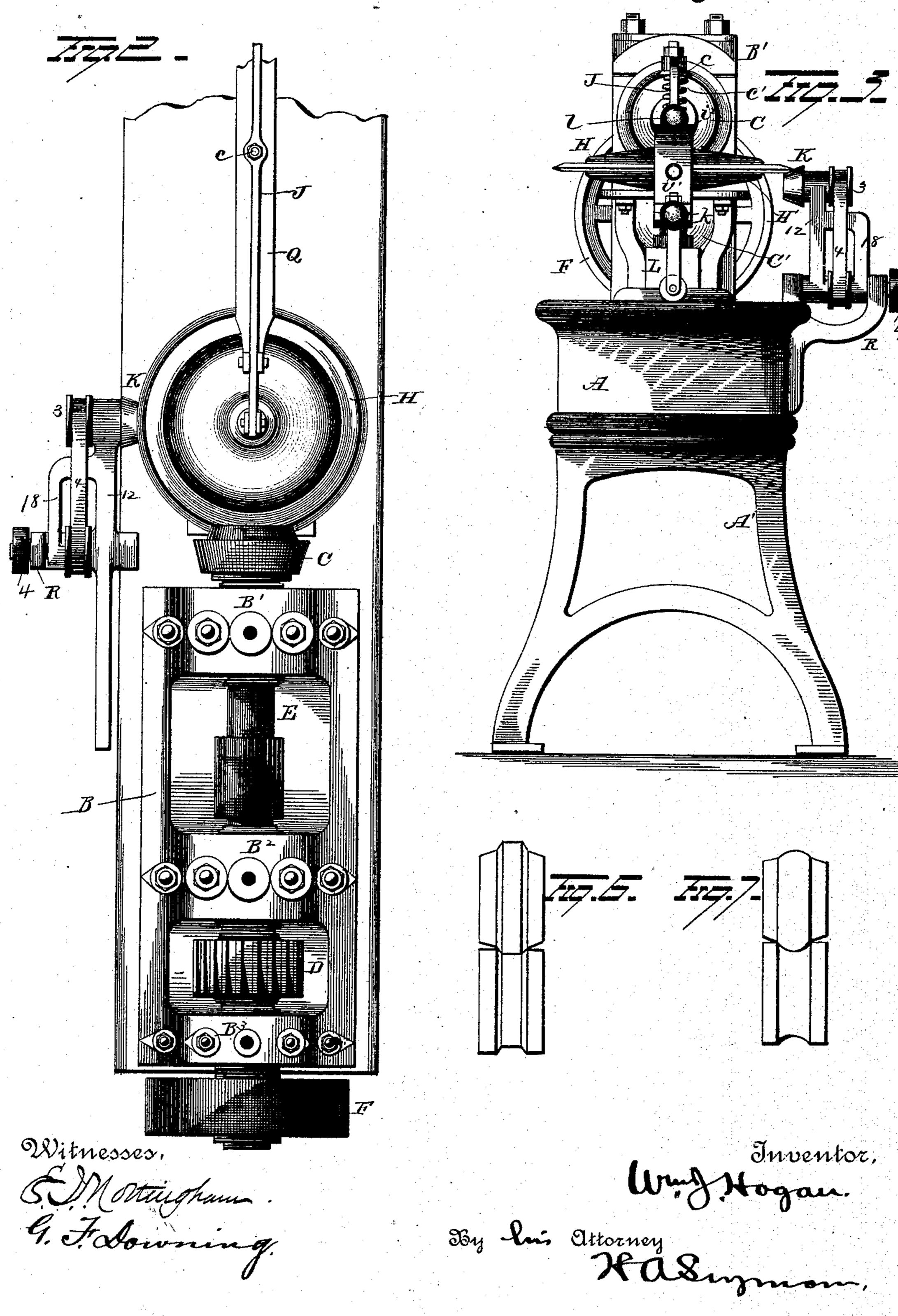
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United States Patent Office.

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MACHINE FOR MAKING DISKS FOR HARROWS.

SPECIFICATION forming part of Letters Patent No. 388,561, dated August 28, 1888.

Application filed May 11, 1888. Serial No. 273,594. (No model.)

To all whom it may concern:

Be it known that I, WILLIAM J. HOGAN, of Johnstown, in the county of Cambria and State of Pennsylvania, have invented certain new and useful Improvements in Apparatus for the Manufacture of Disks for Harrows; and I do hereby declare the following to be a full, clear, and exact description of the invention, such as will enable others skilled in the art to which it appertains to make and use the same.

My invention relates to an improved apparatus for the manufacture of disks for harrows.

In implements of this type a series of circular dished plates of metal are arranged to revolve upon central bearings. These supporting-journals are so curved upon a hollow frame as to present the peripheral edges of the concavo-convex disks at a proper angle to the surface of the ground they are intended to pulverize to insure effective action of the same, the disks being intended to take the place of the teeth of an ordinary harrow.

In order to render the operation of the harrow-disks effective, it is necessary that their edges should be beveled to sharpen them, and it is important that these peripheral edges should be retained in a sharpened condition.

As heretofore constructed, the disks of harrows have been first cut into circular form from a flat plate of rolled metal in a cold condition. These circular blanks are then heated and dished, or rendered concavo convex, by placing them between dies in a drop press or under a drop hammer. They are then finished upon their edges cold by chucking them in a lathe to true up and bevel their edges, thus rendering them of the same diameter and sharpening them.

The method of manufactering harrow-disks as just described involves machine-work that is slow of execution, and therefore expensive. The finished disks are soft on their edges. Consequently they rapidly become dulled in use and are rendered comparatively inefficient in service.

The object of my present invention is to cheapen the cost of dished harrow-disks by more rapid production of the same.

A further object is to provide an improved 50 machine by the use of which the circular blanks of plate metal are rapidly and truly

beveled upon their peripheral edges to sharpen them, and at the same time render the material dense by a rolling compression of said edges during the operation of beveling and 55 sharpening them, the consolidation of the edges thus affected rendering them hard and durable.

With these objects in view my invention consists in an improved construction and combination of parts of a device for the manufacture of disks for harrows which will operate in a rapid and superior manner, as will be hereinafter described, and pointed out in the claims.

Referring to the drawings, Figure 1 represents a side elevation in perspective of my improved machine for roll-swaging the circumference of the harrow disks to consolidate the metal and produce beveled sharp peripheral edges thereon. Fig. 2 is a plan view of the 70 device for beveling the edges of harrow-disks. Fig. 3 is an end elevation of the disk-swaging-machine. Figs. 4 and 5 are detached views of important details. Figs. 6 and 7 are views in section of modified forms of rolls, showing different methods of constructing the same to afford a coniform terminal to one roll and lock the two rolls from lost motion endwise with regard to each other.

A represents the bed-piece or frame of the 80 machine, preferably made of cast-iron, rectangular in form, and provided with supporting-legs A', which are attached upon its under side at each end.

Upon the top surface, at one end of the 85 frame A, a head stock, B, is affixed by bolt-connections. This consists of a base-plate, upon which are integrally mounted the upright brackets or housings B' B² B³. The main housings B' B² are of a suitable height and width, 90 and are located a proper distance apart, so as to admit of the insertion and revoluble support of two shafts, E E', which latter are journaled in boxes adjustably secured in a manner to afford the shafts a correct separation and parallelism of their axes.

At the rear end of the head-stock B the bracket or housing B³ is located. This is of such a height as to admit a single journal-box in which the rearwardly-extended end of the 100 lower shaft, E', is supported and adapted to revolve when actuated by power applied

through a belt-connection of the pulley F, which latter is affixed to the end of the shaft E', projecting outside of the housing B', as

shown in Figs. 1 and 2.

Between the housings B² B³ two spur gearwheels, DD', are located. These are mounted upon and secured to the shafts E E', respectively, and are of such diameter as to permit their teeth to have meshing contact. The other ro ends of the parallel shafts E E' are extended outside of the housing B' and give support to the short cylindrical rolls C C', which are firmly keyed or otherwise affixed in place upon these shafts, respectively as named.

The top roll, C, is rendered true upon its outer surface considered radially from the axis of the shaft E. This surface is diametrically enlarged near its center of length between the points 6 and 7, (see Fig. 1,) and from the point 20 7 to the outer end, 8, the peripheral surface is tapered or rendered slightly conical. The body of the roll C between the shoulders 6 and 7 is tapered slightly toward the housing B', and from the shoulder 6, which is nearest 25 this housing, the roll C is reduced to form a cylindrical hub, 10, which has revoluble contact with the housing-box to which it is adjacent.

The lower roll, C', is provided with a cylin-30 drical portion, C2, which is of the same length as the conically-tapered portion of the upper roll, C, opposite to it, so that the shoulder or square offset produced at the point 7 on these two rolls will have a rolling contact with

35 each other.

From the point 6 to point 8 the lower roll, C', is oppositely tapered to exactly conform to the tapered surface of the upper roll, C, and from point 6 to the inner end of the roll C' 40 the body of this roll is reduced to form a cylindrical hub similar to the hub 10 on the

upper roll.

The revoluble contact of the tapered middle portion of the two rolls CC' and the mov-45 ing abutment of the shoulders formed on these rolls at the point 7 will lock the rolls from lost motion endwise, and, owing to the outer tapered end of the roll C having contact with the cylindrical portion C2 of the lower roll, 50 C', at the point 9, it is apparent that a wedgeshaped opening will be afforded between the surfaces of the ends of the rolls.

Upon the top surface of the frame A of the machine, just in advance of the rolls CC, a 55 saddle-block, L, is mounted and affixed in position. On the end t of this block a cylindrical enlargement is located, it forming the base of the vertically-projecting stud M.

The lever P is provided with an enlarged 60 oblong boss, r, that is perforated vertically at its rear end, this hole being of proper diameter to admit the insertion of the stud M, upon which the lever P may be vibrated laterally. The lever P is furnished with a handle, k, at

65 its free outer end, and also has a depending bracket-arm, i, formed integral with it, this arm i being bifurcated at its lower end to re-

ceive a small wheel, w, adapted to revolve freely between the forked ends of the arm and rest upon the top surface of the bed-plate or 70 frame A of the machine.

A second perforation, p, (see Fig. 4,) is made vertically through the boss r parallel to the stud M, and nearer the outer end of the boss r a hole is formed, it being adapted 75 to receive the standing bolt x. (See Fig. 1, the hole being shown in dotted lines.)

On the top surface of the boss r the lower plate of the mated clamping-flanges H H' is seated. A cross section of these flanges, 80 taken through their center, is shown in Fig. 5. It will be seen that the inner adjacent faces of the flanges H H' are grooved concentrically to reduce the surface of contact.

Both of the flanges are centrally perforated, 85 and the lower one, H', is recessed or counterbored from its top face downward a short distance to allow the collar z, formed integrally with the standing bolt x, to seat itself in this recess. The length of the bolt x is so pro 90 portioned to the thickness of the flange H and boss r, formed on the lever P, that when the parts are assembled and a nut, x', adjusted upon the threaded lower end of the bolt x, so as to bear against the inner shouldered ter- 95 mination of this thread, the flange H' will be held in place, and yet be permitted to rotate freely upon the bolt x as a pivot. The standing bolt x has an integral cylindrical end, y, formed on it, which projects vertically a short 100 distance above the collar z. The central hole made in the top flange, H, is of a size to loosely fit the end y of the bolt x, so that a doweland socket connection is thus produced between the top and bottom circular clamping 105

flanges, HH'.

Upon the surface of the lever P a bracketarm, Q, is mounted. This arm is rigidly secured by bolts i to the lever near its handle k and extends upwardly to a point, i^2 , where 110 it is bent to lie parallel to the lever P and extend forwardly to a point, o, where it is again bent upwardly to engage with its bifurcated end o' the body of the handle-bar J. The bar J extends forward from the end o' to be piv- 115 otally connected to the swivel-bolt h, that enters the central perforation of the top clamping flange, H, which latter has a hub, a, formed on its top surface, that affords a means of connection to the bolt h loosely to this flange H 120 by the insertion of two set-screws, qq, at opposite points, through the side walls of the collar. Their ends will enter a groove formed in the swivel-bolt, and thus permit the flange H to revolve on the swivel-bolt, while it is pre- 125 vented from vertical displacement, so that the vertical vibration of the handle-bar J upon its fulcrum o' will raise the top clamping-flange, H, and clear it from its engagement with the dowel-pin y.

A handle, l, is provided for the bar J, and at a point, c, a spiral spring, c', is introduced and held in place by a bolt, c^2 , the action of the spring having tendency to raise the handle-

bar J and clamp the top flange, H, upon the lower flange, H', or anything introduced between them.

Between the lever P and attached bracket 5 Qa sliding block, s, is located. This is adapted to have a reciprocal movement on its seat, and at its rear edge the spring u is introduced, this spring encircling a bolt, V, that extends from the rear edge of the block s and passes to through the vertical portion of the bracket Q, the projecting end, v', being threaded to receive an adjusting-nut, by which the position of the sliding block s may be adjusted toward or from the clamping-flanges H H', as may be 15 required. The body of the block s is cut away from the front edge to afford space for the introduction and support of the thimbles T T', which are placed upon the bolt T², and thus held loosely to permit their revolution thereon. 2c The adjacent edges of the thimbles T T' are cut away slantingly to produce a V-shaped groove at their point of contact. The relative height of the top face of the lower clampingflange, H', from the supporting surface of the 25 frame A is such in relation to the line of contact between the two rolls C C' that this face will lie in the same horizontal plane with the top surface of the adjacent cylindrical end of the lower roll, C', as shown in Fig. 1. By the 3c provision just mentioned the lower clampingflange, H', will retain a disk or blank clamped upon it in such a horizontal plane as to enter the edge of such a circular plate or disk, if of proper diameter, flatwise between the opposite 35 surfaces of the end of the top and bottom rolls,

 \mathbf{C} \mathbf{C}' . In operation a disk, I, that has been previously heated evenly throughout its body, is introduced between the clamping flanges H 40 H', these flanges having first been moved apart, as shown in Fig. 5, to carry them out of axial line with the rolls CC, which will permit the upper flange to be raised by the handle-bar J. The blank disks I have a central perforation 45 made through them of a proper diameter to permit them to fit the vertical dowel pin y, which is the upper end of the standing bolt x, and on insertion of a disk between the clamping-flanges H H' the pin y is made to enter the 50 hole in the center of the disk. A release of the handle bar J will clamp the two flanges forcibly together by the expanding of the previously-compressed spring c'. The disk I is now in position with its edge protruding concen-55 tric with the peripheries of the clampingflanges H H'. Motion is now communicated to the pulley F, which is rotated in the direction indicated by the arrow on its face. (See Fig. 1.) The revolution of the lower shaft, E', 60 and its attached roll C' will cause the upper shaft, E, and roll C to rotate with equal speed in an opposite direction, so that the rolls C C' move rapidly toward each other, as shown by arrows on their faces. The clamped harrow disk 65 I is now moved laterally by swinging the handle k of the lever P, and the edge and narrow peripheral portion of the body of the disk is

thus brought into contact with the revolving rolls C C'. The position of the grooved thimbles T T' is such with regard to that of the 70 clamped disk I that its edge will bear loosely within the V-groove formed on these thimbles at the apex of its angle, and the strength of the spring u is so proportioned that it will afford a proper support to the block S, and con-75 sequently to the revolving edge of the disk I, when the machine is in operation. As the handle k is held by the operator to keep the edge of the disk I in proper contact with the revolving rolls C C', these rolls will speedily 8c reduce the edge and cause it to conform to the wedge shaped space between them, so as to bevel or taper the same to a thin sharp edge. A rapid rotation of the clamped harrow-disk I, resulting from the enforced introduction of its 85 edge between the rolls, will render the thinned edge uniformly true. The thinning of the edge of the harrow-disk I by the rolls C C' has a tendency to consolidate the same and partially cool it, and when the disk-edges are rolled to go a finish the surface is chilled by the contact of the metal surfaces of these rolls, and a stiff scale is formed, that will resist wear and render the disks more durable.

In order to remove the "fin" edges from 95 the disks while they are being rolled, as just described, a vibrating forked arm, 18, is supported to rock upon the bracket R, which is affixed to the side of the frame A, (see Fig. 3,) and upon the upper end of the arm 12 a short 100 shaft is revolubly mounted. The shaft has a tapering emery-wheel or grindstone, K, attached to its inner end in such a position as to come in contact with the lower side of the revolving disk I, the taper of the emery-wheel 105 or stone K corresponding to the short bevel produced by the lower roll, C', upon the lower surface of the disk at its extreme edge.

A rotation of the emery-wheel or stone K is effected by a pulley, 3, placed on the opposite 110 end of the shaft on which the wheel K is mounted, and a belt (not shown) transmits motion from a lower pulley-shaft mounted in the bracket R and receives motion from some adequate source by means of belt attachment 115 to the pulley 4, mounted on the outer end of the latter-named shaft.

The action of the emery-wheel K, it should be understood, is not intended to grind away any of the surface produced by the rolling 120 compression and consolidation of the edges of the harrow-disks I, but merely to remove accidental inequalities and burrs produced thereon by the contact therewith of the re-enforce thimbles T T', these being necessary to relieve 125 the center hole of strain and steady the action of the rolls.

The concentric grooving of the clamping-faces H H' affords a more secure bite of the plates upon the interposed disks I, and also by 130 reducing the surface of the contact of these faces with the hot disks will prevent an undue absorption of the contained heat of the disks while undergoing the edging process, so that

sufficient heat will remain in the body of the disks after removal from the edging-machine to allow them to be dished under a drop-hammer or in a press by suitable dies without a resewal of heat, as such a renewal would destroy the rolled temper given the edge of the disks, and thus vitiate the good results derived from the rolling consolidation of these edges in the manner hereinbefore set forth.

I do not wish to limit myself to the exact construction of the rolls C C' shown in Figs. 1 and 2, as the form of these rolls may be varied and still obtain the same result in regard to efficiency of action. In Figs. 6 and 7 are 15 shown modified forms of the rolls, which may be employed, if preferred. The view given in Fig. 6 shows the rolls constructed with duplicated ends, so that when one end of the two rolls becomes worn and irregular or rough from 20 hard usage the rolls may be reversed and the opposite ends put into service. Other minor changes of construction might be made without a departure from the spirit of my invention; hence Ldo not wish to limit myself to 25 the exact form shown; but,

Having fully described my invention, what I claim as new, and desire to secure by Letters

Patent, is—

1. The combination, with a set of rolls, of clamping plates and a device for automatically clamping said plates upon a disk, substantially as set forth.

2. The combination, with a set of rolls, of a lever, clamping-plates revelubly supported therein, one of said plates being spring-actuated, and a spring-actuated sliding block adapted to slide in the lever, substantially as set forth.

3. The combination, with a set of rolls, of a laterally-swinging lever, clamping-plates revolubly supported in the short arm of the lever in order that the position of the plates relative to the rolls may be varied, and a spring-actuated block adapted to slide in the lever, said block having rollers therein adapted to bear yieldingly against the periphery of a disk held by the clamping plates, substantially as

set forth.

4. The combination, with a set of rolls, of a lever pivotally supported on a fixed block, a spring-actuated lever pivoted on said lever, and clamping-plates revolubly supported on each of the levers, substantially as set forth.

5. The combination, with a frame, housings mounted on this frame, and boxes adjusted to play in these housings, of two shafts that are geared to revolve toward each other, and a roll-head secured to each shaft, these rolls being formed with duplicate-shaped ends to permit their reversal and the use of either set of the roll-head ends, substantially as set forth.

6. The combination, with a frame, a saddle-block secured to this frame, a lever pivoted upon the saddle-block so as to swing sidewise, and two mated clamping-plates revolubly sup-

ported by the pivoted lever near its fulcrumpoint, of a bracket arm attached to the top face of the swinging lever by one end, and a handlebar pivoted to the other end of this bracketarm and pivotally attached to the upper bifurcated end of a bolt which is swivel-connected to the top clamping-plate by its other or lower end, substantially as set forth.

7. The combination, with a clamping-plate supported to revolve on the end of a lever and 75 having its top face grooved concentrically to reduce its bearing surface, of another clamping-plate similarly grooved and loosely connected to a handle bar by a swivel-bolt, which allows the upper clamping-plate to rotate with 80

the lower end and be vertically moved by a handle-bar, substantially as set forth.

8. The combination, with two rolls supported and adapted to rotate toward each other, the ends of which are formed to roll a beveled 85 edge upon an inserted flat circular metal plate, of a pair of clamping-plates that are held from a lateral displacement of their edges and are adapted to clamp a metal disk between them, and grooved thimbles which are supported to 90 rotate and are by spring action caused to bear against the revolving edge of a clamped disk at a point opposite the axial centers of the rolls, substantially as set forth.

9. The combination, with a frame, a saddle- 95 block, a lever pivoted on the saddle-block and provided with a depending arm and its pivoted roller or wheel that bears upon the upper face of the frame, and a pair of mated clamping-plates that are prevented from edgewise 100 displacement and are adapted to clamp a harrow-disk between them, of a bracket-arm that is attached to the top surface of the pivoted lever, a handle-bar pivoted to the bracketarm, a sliding block, a spiral spring that bears 105 against the sliding block, grooved thimbles revolubly mounted on the sliding block, a swivel-bolt that connects the top clamping. plate with the handle-bar, and a spring which presses the top plate toward the lower plate, 110 substantially as set forth.

10. The combination, with a pair of rolls supported and rotated toward each other, which are shaped to produce a beveled edge on a harrow-disk, of two clamping-plates which readapted to clamp the body of a metal disk concentric with their edges and present its periphery to the adjacent faces of two revolving rolls, and a milling or grinding device that is supported to engage the beveled edge of this readisk and remove slight inequalities from it while it is being rotated, substantially as set forth.

In testimony whereof I have signed this specification in the presence of two subscrib- 125 ing witnesses.

WILLIAM J. HOGAN.

Witnesses:

CYRUS ELDER, SIDNEY POSTLETHWAITE.