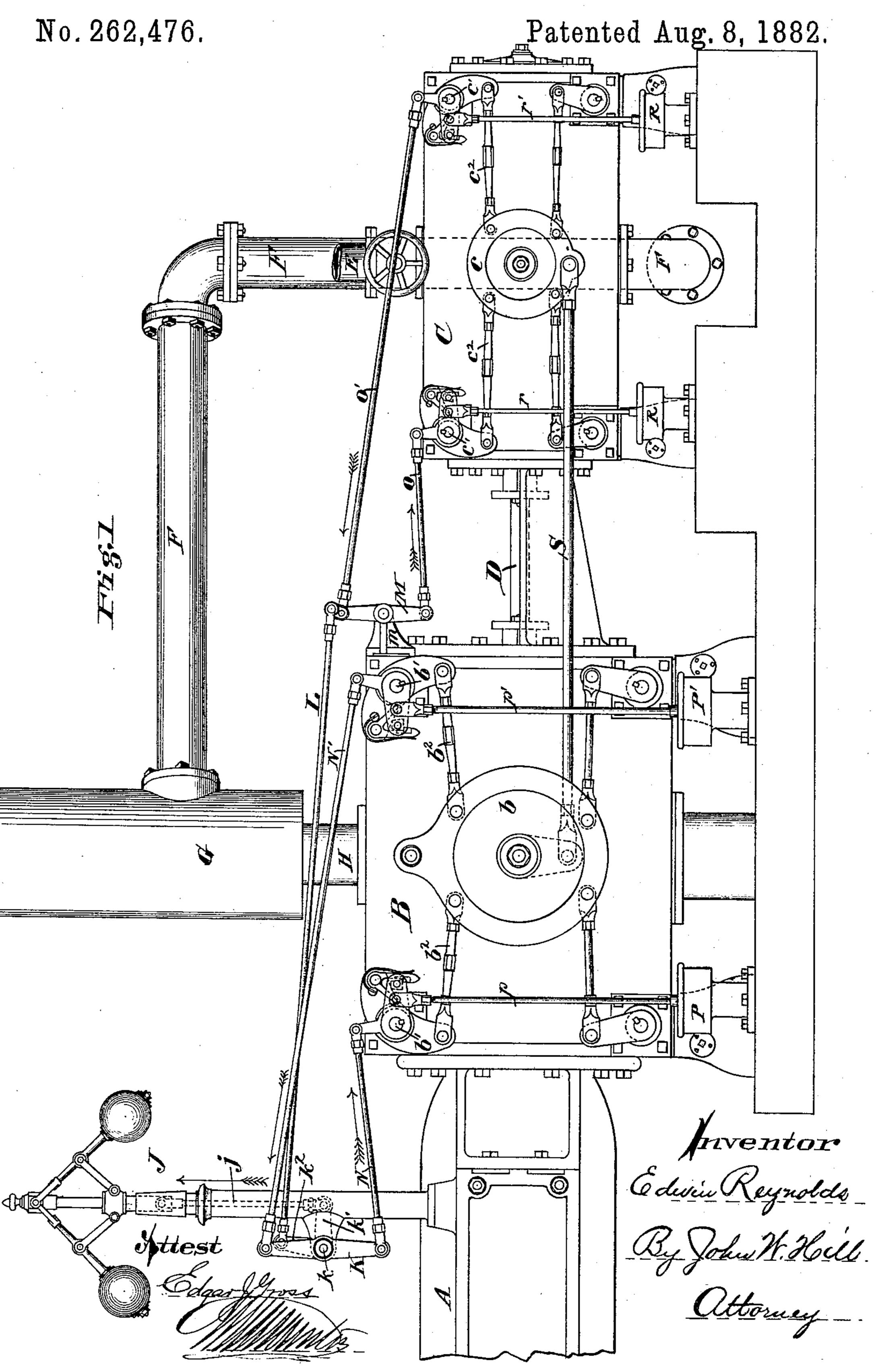
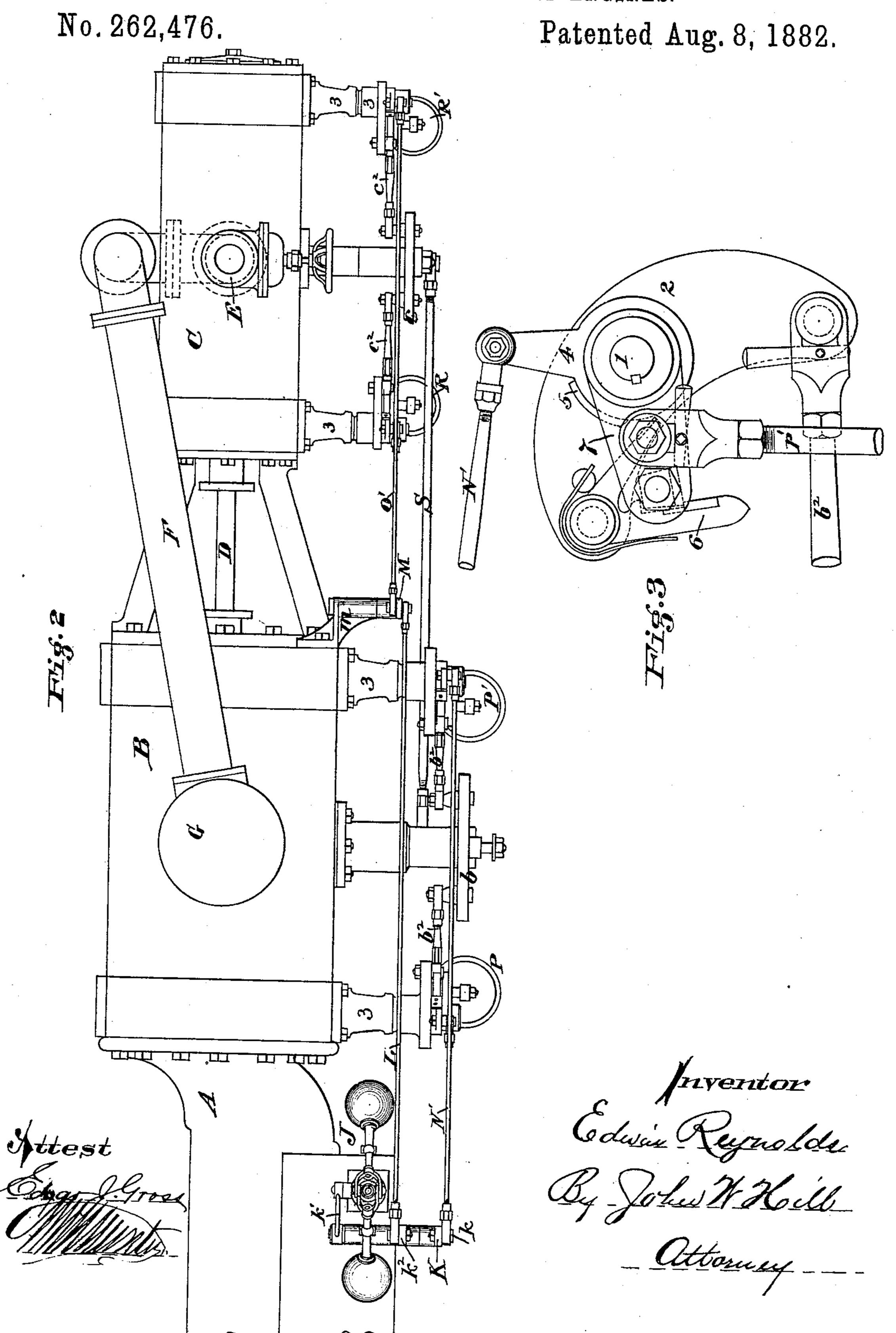
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United States Patent Office.

EDWIN REYNOLDS, OF MILWAUKEE, WISCONSIN.

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SPECIFICATION forming part of Letters Patent No. 262,476, dated August 8, 1882. Application filed April 14, 1882. (No model.)

To all whom it may concern:

Be it known that I, EDWIN REYNOLDS, of Milwaukee, in the county of Milwaukee and State of Wisconsin, have invented certain new 5 and useful Improvements in Valve-Gear for Compound Engines, of which the following is

a specification.

My invention relates to that class of steamengines which consists of two or more steam-10 cylinders, one or more of which are arranged to receive steam direct from the boiler, and one or more of which are arranged to receive steam from the exhaust of the former-mentioned cylinder or cylinders, or from an intermediate re-15 ceiverinto which said exhaust is first delivered. Compound engines, or engines containing two or more cylinders, usually contain one highpressure cylinder, taking steam from the boilers, and one low-pressure cylinder, taking steam 20 from the exhaust of the high-pressure cylinder, or from an intermediate receiver set between the two cylinders and connected to the exhaust-outlet of the high-pressure cylinder and steam-inlet of the low-pressure cylinder. Compound en-25 gines are sometimes constructed with one highpressure steam-cylinder and two low-pressure steam - cylinders, or with two high - pressure steam-cylinders and one low-pressure steamcylinder.

My invention further relates to that variety of compound engines which are automatic in their action—that is, in which the cut-off of steam is controlled by a suitable governor to regulate the speed of engine and proportion 35 the consumption of steam to the load carried.

In the construction of automatic compound engines heretofore the governing or regulating gear has been connected only with the steamvalves of the high-pressure cylinder, the valve-40 motion of the low-pressure cylinder being constant in all its functions. Thus, when the cutoff of the high-pressure cylinder is lengthened or shortened, indicating respectively an increase or reduction of load and diminution or 45 increase of speed, the high-pressure piston, while disposed to promptly restore the proper mean speed and adapt the consumption of steam to the precise demands upon the engine, is resisted by the low-pressure piston, which, 50 with its constant cut-off, is disposed to move

at the increased or retarded speed, and, per con-

sequence, no effect either in regulation of speed or in the consumption of steam can be had from the low-pressure half of the engine, except through the exhaust of the high-pressure 55 cylinder or the intermediate receiver into which the high-pressure cylinder exhausts.

To automatically control the cut-off of both cylinders by connecting the steam-valves thereof with one or two regulators or governors is 60

the object of my invention.

In the accompanying drawings, Figure 1 is an elevation of one style of compound engine known as the "tandem" type, consisting of a high-pressure cylinder mounted behind a low- 65 pressure cylinder, with the pistons of said cylinders secured to a single piston-rod common to both. Fig. 2 is a plan of Fig. 1, and Fig. 3 is a detached view of the steam-valve gear.

Similar letters of reference indicate similar 70

parts.

A is the frame or girder of a steam-engine of the "Corliss" type, one end of which is bolted to the low-pressure steam-cylinder B.

C is the high-pressure steam-cylinder. D is the piston-rod, to which the pistons of

cylinders B and C are secured.

E is the live-steam pipe; e, the stop-valve; F, the exhaust-pipe from the high-pressure cylinder; G, the intermediate receiver, taking 80 the exhaust from the pipe F and delivering through the pipe H to the low-pressure cylinder B.

b is the wrist-plate of the low-pressure cylinder, actuated by the customary eccentric and 85 eccentric-rod from the engine-shaft, and c the wrist-plate of the high-pressure cylinder, actuated by the rod S from the wrist-plate b.

b' b' are the steam-valves of the low-pressure cylinder, worked by the rods b^2 b^2 from the 90 wrist-plate; b and c' c' are the steam-valves of the high-pressure cylinder, worked by the rods c^2 c^2 from the wrist-plate c.

J is the regulator or governor, mounted on the girder A.

K is a double rocker-arm, mounted on the shaftkand actuated by the regulator J through the rod or link j and arm k'. (Shown in dotted lines, Fig. 1.)

L is a rod or link operated from the rock- 100 shaft k by the arm k^2 , and connected to the second double-rocker arm, M, mounted in a

bracket, m, to the rear end of the low-pressure cylinder B.

N N' are the regulating-rods to the low-pressure steam-cylinder valves b' b', and O O' the regulating-rods to the high-pressure steam-

cylinder valves c' c'.

The valve-gear shown in the drawings is of the drop cut-off type, and is clearly detailed in the detached view, Fig. 3, of which 1 is the stem of the well-known Corliss rotative slide-valve; 2, a crescent-shaped rocker-arm pivoted to the bonnet 3, Fig. 2, and actuated by the rods b' b' or c' c'. 4 is an arm pivoted concentric with the valve-stem 1 and rocker-arm 2, which is provided with a circular project-

ing segment, 5, against which "wipes" the inner arm of the hook 6. The arm 4 is actuated by the regulating-rod N. As the arm 5 is rolled to the right or left the hook 6 is released earlier or later in the stroke of the piston, and the steam-valve closed by the vacuum dashpot P P' or R R', acting through the rods p p' or rr' and arm 7, keyed to the valve-stem 1.

The peculiar form of valve-detaching gear shown is not material to my invention. Indeed, it is equally applicable to that class of automatic engine which contains a steam-valve permanently connected with a variable eccen-

tric or with an adjustable link.

By reference to Fig. 1 the arrows indicate the direction in which the rods or links L, N N', and O O', and j move to shorten the cut-off, and of course motion in an opposite direction has the effect to lengthen the cut-off.

of both cylinders of a compound engine for maximum economy shall be subject to governor control in order that the speed shall be practically constant, and that the consumption of steam shall be proportioned to the demands of the load—conditions which cannot be at-

tained with one set of steam-valves (high pressure cylinder) only, subject to governor control, and the other set (low-pressure cylinder) having an inversible metion

ing an invariable motion.

In the type of compound engine shown in the drawings, one governor can be arranged to control the cut-off of steam to both cylinders; but with the two cylinders set parallel and separated some distance to enable the pistons to connect with cranks at quarters on the engine-shaft, two governors, one for the steamvalves of each cylinder, may be found preferable.

Having described my invention, what I claim 55

1. The combination of the steam-valves of the high and low pressure cylinders and an intermediate receiver which receives the exhaust-steam from the high-pressure cylinder 60 and delivers it to the low-pressure cylinder, with automatic governor mechanism for regulating the steam-valves of both cylinders and causing them to simultaneously cut off earlier or later in the strokes of the piston, substantially as 65 described.

2. The combination, with the gears giving motion to the steam-valves of the two cylinders of a compound engine, of the regulator J, rocker-arms K and M, and links or rods L, N 70 N', and O O', arranged to simultaneously lengthen or shorten the cut-off or closing movement of said valves, substantially as de-

scribed.

Intestimony whereof I have signed my name 75 to the foregoing specification in the presence of two subscribing witnesses.

EDWIN REYNOLDS.

Witnesses:

G. H. TUCKER, Jr., G. M. HINKENY.